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ATHENA: Aero-Thermo-Elastic Nonlinear reduced order modeling for hypersonic Airframes

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Aero-THermo-Elastic Nonlinear reduced order modeling for hypersonic Airframes: ATHENA

Morteza Karamooz Mahdiabadi, Paolo Tiso

**Final Report** 

# **Motivation**





## Assess the structural dynamics in presence of time-varying thermal and aerodynamic loads, for long times: fatigue analysis

AFOSR – Grant No. 1191404 – Reduced Order Modeling for Hypersonic Aeroelasticity: **ROMA** | Jan 2016-Dec 2018

AFOSR – Grant No. FA9550-18-1-0508 - Aero-THermo-Elastic Nonlinear reduced order modeling for hypersonic Airframes: **ATHENA** | Sept 2019-Sept 2021

# Motivation – what model features we need?

- Arbitrary geometries: large FEM
- Temperature field space and time dependent
- Geometric nonlinearities to account for
  - bending stiffness reduction due to thermally induced compression loads
  - Buckling/snapthrough
  - Large deflections
  - Limit cycle oscillations
- Temperature dependent material properties degradation
- Aero loads via piston-theory + fluctuations due to turbulence
- Aero-thermo-mechanical coupling

# Motivation – where is the bottleneck?

- Design/optimization require lots of analyses
- High Fidelity Models (HFMs) are unfeasible do to time and memory resources required (1 iteration might require hours/days)
- Current practice: neglect coupling/transient response, or resort to lumped models to trade for speed

Need for Reduced Order Models (ROMs)

**Outcome**: ROMs that enable design iterations at reasonable times (order of magnitude faster than HFMs), while **keeping all the essential features of the HFMs**, without requiring abstraction, simplification or lumping.

# One-way thermo-elastic coupling

1. S. Jain, <u>P. Tiso</u>, Journal of Sound and Vibration, (2020) [PDF]

Thermal problemStructural problem $\mathbf{M}_{T}\mathbf{T}' + \mathbf{f}_{T}(\mathbf{T}) = \mathbf{h}(\tau)$  $\mathbf{T}(\tau)$  $\mathbf{M}\ddot{\mathbf{u}} + \mathbf{C}\dot{\mathbf{u}} + \mathbf{f}(\mathbf{u}, \mathbf{T}) = \mathbf{g}(t)$ 

Solve the thermal problem first, then pass the  $\mathbf{T}(\mathbf{x}, \tau)$  to the structural problem.

Non-existence of

- 1. equilibrium
- 2. invariant spaces for reduction

Standard techniques (hot modes, cold modes, etc),not well grounded

Temperature evolves slowly when compared to structural periods: **time scale dichotomy**.

# **Method of Multiple Scales**

Solution depends on two different time scales

$$\mathbf{u} = \mathbf{u}(t,\tau)$$

Expansion in  $\epsilon$ 

$$\mathbf{u}(t,\tau) = \mathbf{u}_0(t,\tau) + \epsilon \mathbf{u}_1(t,\tau) + \epsilon^2 \mathbf{u}_2(t,\tau) + \dots$$

Leading order: nonlinear problem

$$\mathcal{O}(1): \mathbf{M} \frac{\partial^2 \mathbf{u}_0}{\partial t^2} + \mathbf{C} \frac{\partial \mathbf{u}_0}{\partial t} + \mathbf{f} (\mathbf{u}_0, \mathbf{T}(\tau)) = \mathbf{p} (t, 0)$$
  
Linear system with slow temperature variation:  $\tau$  is a parameter  
$$\mathbf{M} \frac{\partial^2 \mathbf{u}_0}{\partial t^2} + \mathbf{C} \frac{\partial \mathbf{u}_0}{\partial t} + \mathbf{K} (\mathbf{T}(\tau)) \mathbf{u} + \mathbf{b} (\mathbf{T}(\tau)) = \mathbf{p} (t, 0)$$

This justifies parametric equilibrium... and reduction basis.  $\mathbf{u}_{eq}(\tau) = -\mathbf{K} \left(\mathbf{T}(\tau)\right)^{-1} \mathbf{b}(\mathbf{T}(\tau)) \qquad \left[\mathbf{K}(\mathbf{T}(\tau)) - \omega_i^2(\tau) \mathbf{M}\right] \boldsymbol{\phi}_i(\tau) = \mathbf{0}$ 

## Modal basis variation



Modes might experience **veering** as temperature changes. This can be taken into account by the algorithm discussed in [4].

# **Galerkin projection**

$$\boldsymbol{\Phi}(\tau)^{T} \mathbf{M} \boldsymbol{\Phi}(\tau) \frac{\partial^{2} \mathbf{q}_{0}}{\partial t^{2}} + \boldsymbol{\Phi}(\tau)^{T} \mathbf{C} \boldsymbol{\Phi}(\tau)^{T} \frac{\partial \mathbf{q}_{0}}{\partial t} + \boldsymbol{\Phi}(\tau)^{T} \mathbf{f} \left( \mathbf{q}_{eq}(\tau) + \boldsymbol{\Phi}(\tau) \mathbf{q}_{0}, \mathbf{T}(\tau) \right) = \boldsymbol{\Phi}(\tau)^{T} \mathbf{p} \left( t, 0 \right)$$

- $\checkmark$  The ROM adapts to the underlying, slowly changing equilibrium;
- $\checkmark$  No basis time derivatives present;





- Temp. profile traveling over the arch
- Mechanical load excites first 3 modes
- ROM: 5 modes, interpolated between 19 temperature configurations

## **Example: Curved Arch**



# **Parametric ROM**

$$\boldsymbol{\Phi}(\tau)^{T} \mathbf{M} \boldsymbol{\Phi}(\tau) \frac{\partial^{2} \mathbf{q}_{0}}{\partial t^{2}} + \boldsymbol{\Phi}(\tau)^{T} \mathbf{C} \boldsymbol{\Phi}(\tau)^{T} \frac{\partial \mathbf{q}_{0}}{\partial t} + \boldsymbol{\Phi}(\tau)^{T} \mathbf{f} \left( \mathbf{q}_{eq}(\tau) + \boldsymbol{\Phi}(\tau) \mathbf{q}_{0}, \mathbf{T}(\tau) \right) = \boldsymbol{\Phi}(\tau)^{T} \mathbf{p} \left( t, 0 \right)$$

The highlighted terms need to be efficiently computed.

How to make the ROM efficient online?

- 1. Non-intrusive computation of ROM terms
- 2. Efficient interpolation

## Non-intrusive methods

$$\begin{split} \tilde{\mathbf{M}} \ddot{\mathbf{q}} + \tilde{\mathbf{C}} \dot{\mathbf{q}} + \tilde{\mathbf{K}} \mathbf{q} + \tilde{\mathbf{f}}_{nl}(\mathbf{V}\mathbf{q}) &= \tilde{\mathbf{g}} \\ \\ & \mathbf{F} \text{unction of modal coordinates only!} \\ \mathbf{V}^T \mathbf{f}_{nl}(\mathbf{V}\mathbf{q}) \stackrel{\Rightarrow}{=} (\tilde{\mathbf{f}}_{nl})_I &= \sum_{i}^{m} \sum_{j}^{m} \alpha_{ijI} q_i q_j + \sum_{i}^{m} \sum_{j}^{m} \sum_{k}^{m} \beta_{ijkI} q_i q_j q_k \end{split}$$

Coefficients of **assumed nonlinear force** obtained by **fitting** with nonlin. forces necessary to hold structure into the shape of dominant modes, **at chosen amplitudes**.

Done conveniently with off-the shelf FE programs (Abaqus, Ansys, ...): "non-intrusive".

## Implicit Condensation and Expansion (ICE)

M. I. McEwan, J. R. Wright, J. E. Cooper, and A. Y. T. Leung, "A combined modal/finite element analysis technique for the dynamic response of a non-linear beam to harmonic excitation," Journal of Sound and Vibration, vol. 243, pp. 601-624, 2001.

## Enforced Displacements (ED)

A. A. Muravyov and S. A. Rizzi, "Determination of nonlinear stiffness with application to random vibration of geometrically nonlinear structures," Computers & Structures, vol. 81, pp. 1513-1523, 2003

# Non-intrusive methods

State of the art:

- 1. They require lots of static solutions/evaluations
- 2. The modes necessary to capture the geometric nonlinearity (**Dual modes**) in the reduction basis also require extensive computations



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Our contribution:

**Non-intrusive Modal derivatives** appended to the linear modes basis M.K. Mahdiabadi, <u>P. Tiso</u>, A. Brandt, DJ Rixen, **MSSP** (2020) [PDF]

$$\mathbf{K}_{t}\left(\mathbf{u}_{eq}(\tau),\mathbf{T}(\tau)\right)\boldsymbol{\theta}_{ij}=-\left[\frac{\partial^{2}\mathbf{f}}{\partial\mathbf{u}\partial\mathbf{u}}\left(\mathbf{u}_{eq},\mathbf{T}(\tau)\right)\cdot\boldsymbol{\phi}_{j}\right]\boldsymbol{\phi}_{i}$$

Modal derivatives automatically account for geometrically nonlinear deformation induced by vibration modes. Their computation is systematic, as compared to Dual Modes, and lead to generally more consistent results.



# Non-intrusive methods

Our contribution (cont'd):  $\mathbf{u} = \mathbf{\Xi}(\mathbf{q}) = \mathbf{\Phi}\mathbf{q} + \frac{1}{2}\mathbf{\Theta}\mathbf{q}\mathbf{q}, \ \mathbf{\Xi}: \mathbb{R}^m \to \mathbb{R}^n$ 

Usage of **quadratic manifold** (built with VMs and MDs) to significantly reduce the offline cost of training cases.

## Example: flat beam







Frequency [Hz] DISTRIBUTION A: Distribution approved for public release. Distribution unlimited

# Interpolated ROM



Each support point generates a **non-intrusive nonlinear** ROM  $^t$ 

$$\dot{ ilde{\mathbf{z}}} = \mathbf{A}_i ( ilde{\mathbf{z}} - ilde{\mathbf{z}}_i) + \widetilde{\mathbf{f}}_i ( ilde{\mathbf{z}} - ilde{\mathbf{z}}_i) + \widetilde{\mathbf{p}}$$

 $\mathbf{\hat{M}}\mathbf{\ddot{q}}(t) + \mathbf{\hat{K}}^{(1)}\mathbf{q}(t) = \mathbf{\hat{f}}(t)$ 

 $W_{ij} = \int_0^T \left| q_i(t) q_j(t) \right| dt$ 

ROMs are weighted online by radial basis functions

$$\dot{\tilde{\mathbf{z}}} = \sum_{i} w_i (\tilde{\mathbf{z}} - \tilde{\mathbf{z}}_i) \left[ \mathbf{A}_i (\tilde{\mathbf{z}} - \tilde{\mathbf{z}}_i) + \tilde{\mathbf{f}}_i (\tilde{\mathbf{z}} - \tilde{\mathbf{z}}_i) + \tilde{\mathbf{p}} \right]$$

## **Example: curved shell**

Temperature distribution: uniform in space, increasing in time

ROM: 9 modes, interpolated between
25 temperature configurations





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## **Example: curved shell**

1. MK Mahdiabadi, P. Tiso, under preparation



# Summary

- Slow variation in temperature is not only physically relevant but also essential to justify model reduction using an adaptive reduction basis.
- Use of multiple scales method to exploit the time scale dichotomy.
- Order epsilon correction important in presence of small mechanical loads.
- In the geometrically nonlinear setting, the basis is enriched using the modal derivatives corresponding to significant vibration modes.
- Non-intrusive construction of ROM using modal derivatives and quadratic manifolds for increased offline efficiency
- For hyper-reduction of nonlinear terms, it is possible to avoid HFM based training by lifting modal solutions on quadratic manifolds.
- Online ROM interpolation for efficiency once precomputed equilibria and basis at support points are available.

# **Remaining Technical Challenges**

- Inclusion of aerodynamic and turbulence
  - Prof. Dimitris Drikakis (un. Cyprus) will develop an acoustic model which will be validated against high-fidelity numerical simulations and experiments, and will offer the model to ETH for inclusion into the acoustic-thermo-elastic analysis framework for hypersonic airframes.
  - Thermal buckling/snap-through?
- Parametrization
  - Recently developed a ROM that includes geometric uncertanties (J. Marconi, P. Tiso, F. Braghin, CMAME, 2019)
  - An idea for slightly curved structures?
- Efficient online interpolation for reduced operators
  - Trajectory Piecewise weighted Linearization

## **Publications**

- 1. MK Mahdiabadi, P. Tiso, under preparation
- 2. MK Mahdiabadi, P. Tiso, A. Brandt, DJ Rixen, MSSP (2020)
- 3. MK Mahdiabadi, A Bartl, D Xu, P Tiso, DJ Rixen, Journal of Sound and Vibration (2019)
- 4. S. Jain, P. Tiso, Journal of Sound and Vibration, (2020)
- 5. S. Jain, P. Tiso, ASME Journal of Computational and Nonlinear Dynamics (2019)
- 6. S. Jain, P. Tiso, G. Haller, Journal of Sound and Vibration (2018)
- 7. S. Jain, P. Tiso, ASME Journal of Computational and Nonlinear Dynamics (2018)
- 8. J.B. Rutzmoser, D.J. Rixen, S. Jain, P. Tiso, Computers & Structures (2017)
- 9. S. Jain, P. Tiso, J.B. Rutzmoser, D.J. Rixen, Computers & Structures (2017)