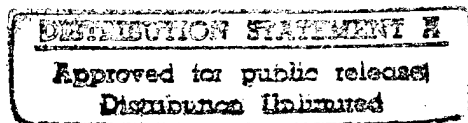


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NOTICE

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1 Navy Case No. 77348

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AN INSERT FOR AN OPENING IN

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A COMPOSITE MATERIAL PRESSURE VESSEL

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STATEMENT OF GOVERNMENT INTEREST

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The invention described herein may be manufactured and used
by or for the Government of the United States of America for
governmental purposes without the payment of royalties thereon or
therefor.

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BACKGROUND OF THE INVENTION

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(1) Field of the Invention

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The invention relates to pressure vessels, and is directed
more particularly to an insert which fills an opening in a wall
of such a vessel, frames a smaller aperture, and serves as a
mounting for an aperture closure member, such as a hatch or
acoustical device.

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(2) Description of the Prior Art

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The United States Navy originally developed a deep diving
shell in the warhead section of a practice torpedo. Such shells
typically contain transducers and recorders which provide a

21

22

1 record as to the performance of a torpedo in a trial or test,
2 such as a simulated attack, or the like.

3 The shell is cylindrically-shaped and is provided with a
4 layered wall having a core of an aluminum honeycomb structure,
5 bounded by inboard and outboard composite skins of glass fiber
6 reinforced epoxy. The honeycomb core is about 1.1 inch thick and
7 the composite skins are each about .1 inch thick. When it is
8 desired to provide an opening in the shell wall for access or for
9 mounting of a transducer, or the like, it is necessary to provide
10 a frame having an aperture therein and having means for
11 supporting an access hatch, or an instrument, and for providing
12 strength to the aperture defined thereby. The honeycomb shell
13 wall structure lacks the strength to support an aperture therein.

14 Accordingly, an insert typically is provided for filling the
15 opening in the shell wall, the insert having sufficient strength
16 to support an aperture therein and a selected closure member.

17 The inserts used heretofore have generally been of aluminum.

18 When aluminum inserts are used with shells having wall skins of
19 glass-epoxy composites, the modulus of elasticity of the insert
20 exceeds the modulus of elasticity of the wall skins by over 400%
21 in a direction through the thickness of the skins. Failures have
22 resulted from the mismatch of moduli of elasticity. A problem
23 that one encounters in attempting to correct such imbalance, is

1 the fact that the shell skins have widely varying moduli of
2 elasticity in directions through the thickness, longitudinally,
3 and through the "hoop", that is, through the curve of the wall
4 skin. It has been found difficult to provide an insert having a
5 modulus of elasticity substantially equal to the shell skin
6 moduli in all three directions.

7 Thus, there is a need for an insert having a modulus of
8 elasticity similar to the "through the thickness" modulus of
9 elasticity of the shell wall skins and relatively close to the
10 transverse, or "hoop", and longitudinal moduli of elasticity of
11 the shell wall skins.

12 Inserts are also necessary in conventional pressure vessels
13 containing a pressurized gas or fluid. The one difference is
14 that the pressurized environment is on the interior of the
15 pressure vessel wall. Similar problems occur with openings in
16 these walls.

17

18 SUMMARY OF THE INVENTION

19 It is, therefore, an object of the invention to provide an
20 insert for a pressure vessel, the insert having sufficient
21 strength to support an aperture therein and having a modulus of
22 elasticity reasonably close to three moduli of elasticity present

1 in the walls of the vessel and extending in three different
2 directions.

3 With the above and other objects in view, as will
4 hereinafter appear, a feature of the present invention is the
5 provision of an insert for an opening in a pressure vessel wall
6 having a central core sandwiched between inboard and outboard
7 skins. The insert is adapted to support a closure member. The
8 insert comprises a compression molded composite plate having an
9 outboard surface and an inboard surface interconnected by outer
10 edge surfaces, and defining an aperture extending therethrough
11 from the plate outboard surface to the plate inboard surface.
12 The plate is provided with a modulus of elasticity, in a
13 direction through the thickness of the plate, which is within
14 about 30% of a first modulus of elasticity of the pressure vessel
15 wall inboard and outboard skins in directions through the
16 thicknesses of the skins. The plate modulus of elasticity is
17 further within about 30% of a second modulus of elasticity of the
18 wall inboard and outboard skins in a widthwise, or "hoop",
19 direction. The plate modulus of elasticity is still further
20 within about 15% of a third modulus of elasticity of the wall
21 inboard and outboard skins in a longitudinal direction.

22 The above and other features of the invention, including
23 various details of construction and combinations of parts, will

1 now be more particularly described with reference to the
2 accompanying drawings and pointed out in the claims. It will be
3 understood that the particular device embodying the invention is
4 shown by way of illustration only and not as a limitation of the
5 invention. The principles and features of this invention may be
6 employed in various and numerous embodiments without departing
7 from the scope of the invention.

8

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BRIEF DESCRIPTION OF THE DRAWINGS

10 Reference is made to the accompanying drawings in which is
11 shown an illustrative embodiment of the invention, from which its
12 novel features and advantages will be apparent.

13 In the drawings:

14 FIG. 1 is a perspective view of a wall portion of a pressure
15 vessel with an insert fixed thereto; the insert being
16 illustrative of an embodiment of the invention;

17 FIG. 2 is an enlarged perspective view of a plate portion of
18 the insert of FIG. 1; and

19 FIG. 3 is a transverse sectional view showing the insert in
20 place in a portion of a pressure vessel.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Referring to FIG. 1, it will be seen that a pressure vessel 10 includes a cylindrically-shaped wall 12 having a honeycomb core 14 of metal, usually aluminum, sandwiched between an inboard composite skin 16 and an outboard composite skin 18. The composite skins 16, 18 are typically of fiber glass-reinforced epoxy and are each about .1 inch thick. The honeycomb core 14 is about 1.1 inches thick. The shell wall 12 is provided with an opening 20 for the mounting of an access hatch, a transducer or other instrument, or other closure member (not shown).

An insert 22 (FIGS. 1-3) is inserted between inboard composite skin 16 and outboard composite skin 18 and into the opening 20 and fixed therein. The insert 22 includes a plate 24 having an outboard surface 26 and an inboard surface 28 interconnected by outer edge surfaces 30, 32, 34 and 36 (FIG. 2) and defining an aperture 38 extending therethrough from plate outboard surface 26 to plate inboard surface 28.

The plate 24 is compression molded with a curvature equal to the curvature of shell wall 12. The plate 24 is of a non-metallic composite material, preferably a glass-reinforced epoxy having glass fibers (preferably "S2" fibers) randomly dispersed therein. The plate 24 is of a thickness of about 1.1 inches, is provided with a modulus of elasticity of about 2.3 million

1 p.s.i., a density of about 0.066 lbs./ in³, and a compressive
2 yield strength of up to about 35,000 p.s.i.

3 In the shell wall inboard and outboard skins 16, 18, the
4 reinforcing glass fibers are oriented in a ± 540 pattern relative
5 to the longitudinal direction of the shell. Accordingly, the
6 skins 16, 18 exhibit a different modulus of elasticity in the
7 "through the thickness" direction, (about 1.8 million p.s.i.) the
8 longitudinal direction (about 2.66 million p.s.i.) and the "hoop"
9 direction (about 3.3 million p.s.i.). The modulus through the
10 thickness is of greatest importance for this application. Thus,
11 the modulus of elasticity of plate 24 is about 30% greater than
12 the "through the thickness" modulus of skins 16, 18, about 15%
13 less than the longitudinal modulus of skins 16, 18, and about 30%
14 less than the "hoop" modulus of skins 16, 18. The modulus of
15 elasticity for aluminum inserts is about 10 million p.s.i., or
16 550% greater than the shell skins "through the thickness"
17 modulus, about 376% greater than the shell skins longitudinal
18 direction modulus, and about 300% greater than the shell skins
19 "hoop" direction modulus.

20 The insert further includes a continuation of the shell wall
21 outboard skin 18 covering and fixed to the plate outboard surface
22 26 and a continuation of the shell wall inboard skin 16 covering
23 and fixed to the plate inboard surface 28. The plate 24 is of a

1 thickness of about 1.1 inch, corresponding to the thickness of
2 the shell wall core 14, and the plate skins 40, 42 are each of a
3 thickness of about .1 inch, corresponding to the thicknesses of
4 the shell wall skins 18, 16. Thus, the juncture of shell wall 12
5 and insert 22 provides a smooth continuous surface inboard and
6 outboard.

7 Referring to FIG. 3, it will be seen that the composite
8 plate 24 is provided, as by machining, with a suitable
9 configuration for the closure to be mounted thereon. For
10 example, the plate shown in FIG. 3 is configured to mount an
11 acoustic transducer (not shown). The plate 24 is provided with a
12 aperture 38 by which electrical components, including a "pinger"
13 are inserted into shell 10, and is provided with a recess 44 for
14 receiving the transducer and backplate portions, and any
15 protective grid and appropriate seals (not shown). Obviously,
16 recess 44 can be positioned facing the interior of the vessel for
17 applications requiring that construction.

18 There is thus provided an insert which may be formed of the
19 same thickness as the pressure vessel shell wall, which is of
20 sufficient strength to support an aperture therein, and which
21 exhibits a modulus of elasticity relatively close to the through
22 the thickness modulus of the pressure vessel shell wall in which
23 the insert is disposed.

1 There is further provided an insert which is about one-third
2 lighter than the usual aluminum insert.

3 There is still further provided an insert which is
4 susceptible to easy changes in manufacture which result in an
5 insert suited for use in conjunction with other than fiber glass-
6 reinforced epoxy shell wall skins. If a different skin is used
7 on the shell, the fiber volume of the glass in the compression
8 molded epoxy can be varied to match a modulus of elasticity of
9 the skin.

10 It is to be understood that the present invention is by no
11 means limited to the particular constructions herein disclosed
12 and/or shown in the drawings.

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6 ABSTRACT OF THE DISCLOSURE

7 A pressure vessel insert comprises a plate having an
8 outboard surface and an inboard surface interconnected by outer
9 edge surfaces, and defining an aperture extending therethrough
10 from the outboard surface to the inboard surface. The plate is
11 provided with a modulus of elasticity within about 30% of a first
12 modulus of elasticity of inboard and outboard skins of a wall
13 portion of the pressure vessel shell around an opening in which
14 the insert is disposed, in directions through the thickness of
15 the skins.

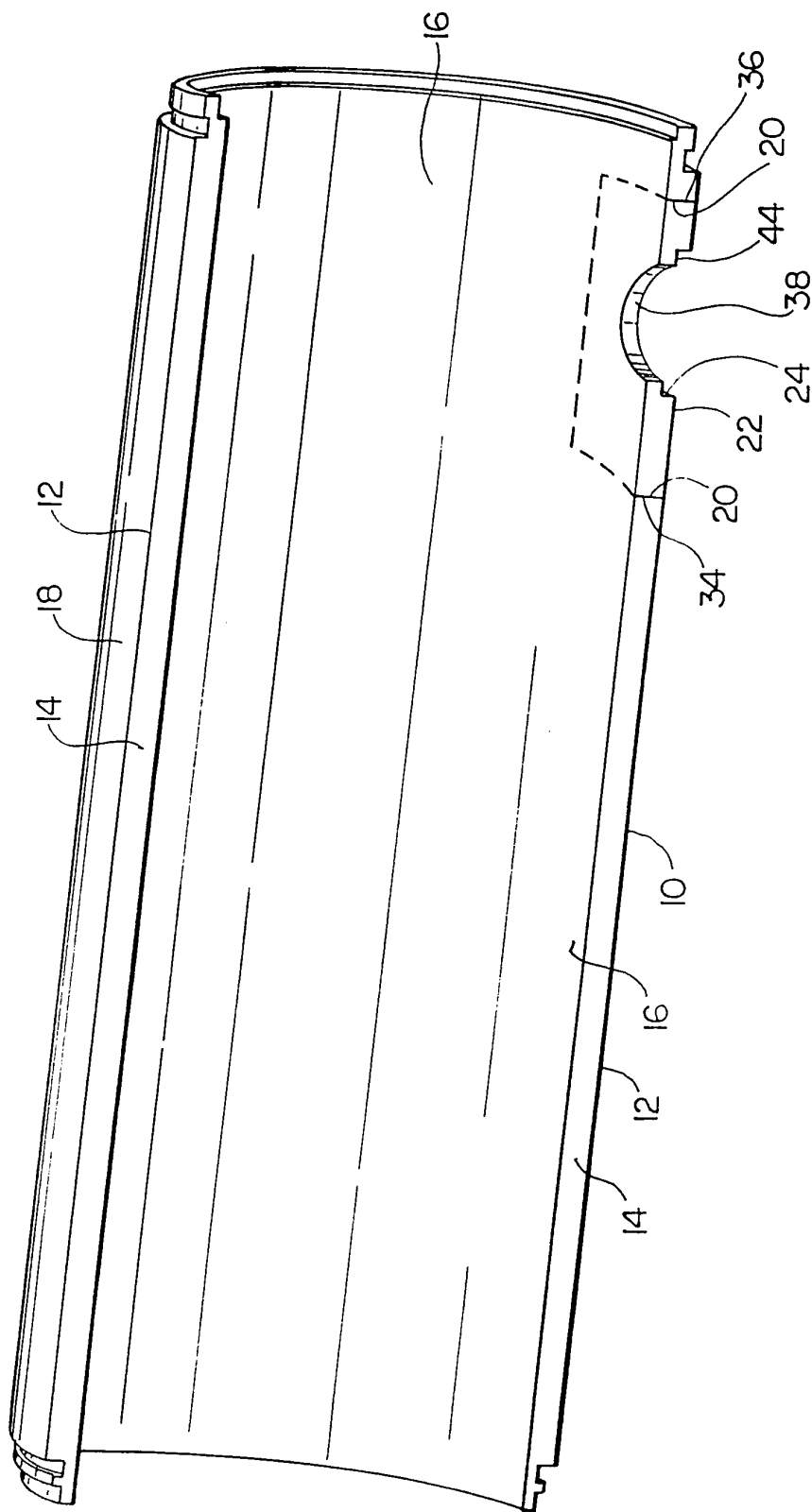


FIG. 1

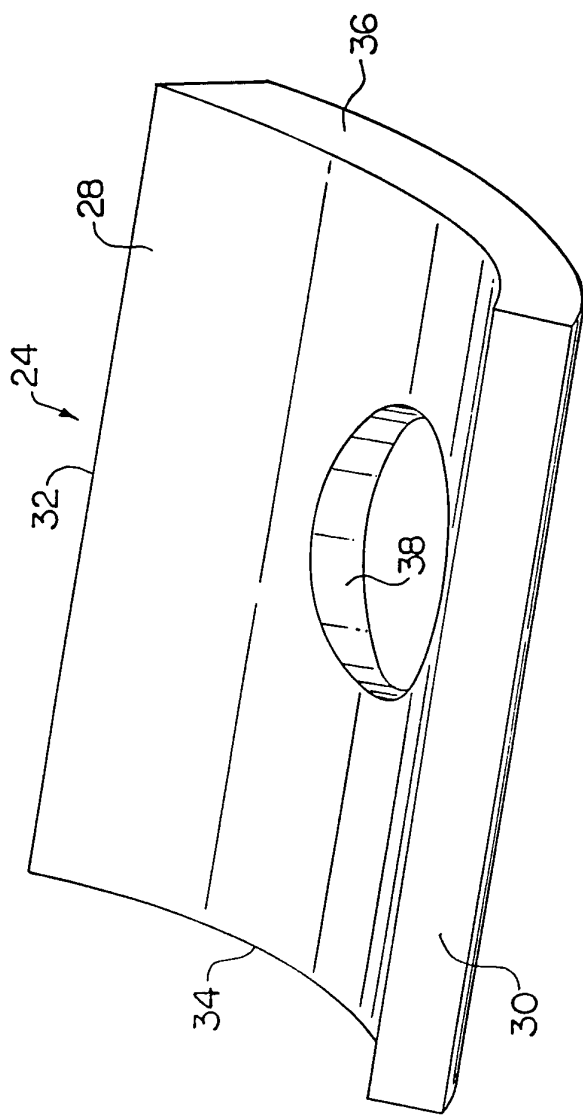


FIG. 2

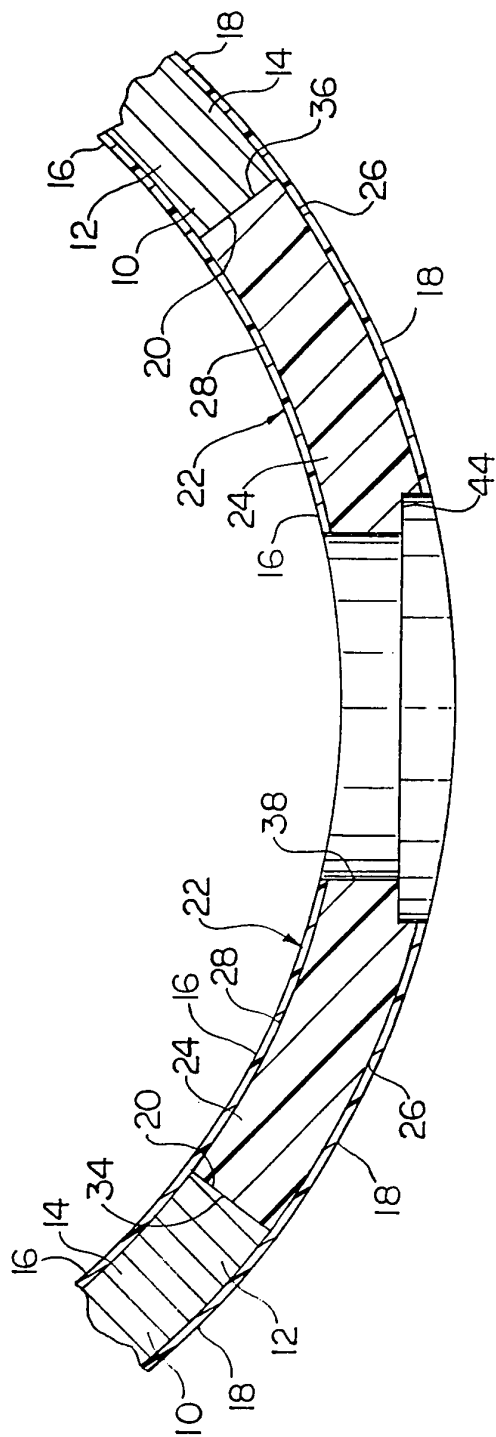


FIG. 3