

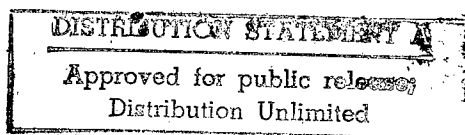
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1 Navy Case No. 77951

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3 STRUT-MOUNTED DRAG BALANCE

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5 STATEMENT OF GOVERNMENT INTEREST

6 The invention described herein may be manufactured and used
7 by or for the Government of the United States of America for
8 governmental purposes without the payment of any royalties
9 thereon or therefore.

10
11 BACKGROUND OF THE INVENTION

12 (1) Field of the Invention

13 The present invention relates generally to the measurement
14 of drag forces on a surface, and more particularly to a strut-
15 mounted drag balance for measuring drag force on a flat plate
16 moving through a medium.

17 (2) Description of the Prior Art

18 The performance of a vehicle moving through a medium is
19 limited by the drag force of the medium on the vehicle. To
20 improve efficiency and reduce the power required to move the
21 vehicle through the medium, numerous drag reduction methods have
22 been tried. It is customary to test drag reduction methods by
23 model testing in towing basins or wind and water tunnels where a

1 scaled model of the vehicle is towed through the medium or the
2 medium is flowed past the model. To measure drag forces on the
3 model, a towing post, or strut is attached to the model through a
4 load cell, or drag balance. Typical prior art drag balances, as
5 exemplified by U.S. Patent No. 5,343,742 to Cusanelli et al.,
6 consist essentially of a hollow metal cube, generally four inches
7 on a side, with portions of the sides removed, leaving planar top
8 and bottom surfaces connected by four legs. The strut is
9 attached to the top surface and the model is attached to the
10 lower surface. Forces exerted on the model cause bending in the
11 legs which is measured using strain gages attached to the legs.
12 When the model is submerged in the medium such that the towing
13 post extends into the medium, the strut is streamlined to
14 minimize extraneous drag forces exerted on the strut, and the
15 drag balance is placed within the scaled model. Testing of drag
16 reduction methods which do not rely on vehicle shape to reduce
17 drag, e.g., polymer injection and magneto hydrodynamic turbulence
18 control devices, is complicated by the difficulty of predicting
19 drag on a model shaped as a scaled vehicle. It is preferable to
20 test such drag reduction methods using a flat plate model since
21 drag calculations are much simpler for a flat plate model than
22 for a hydrodynamically shaped model, thus actual test results can
23 be easily compared to theoretical predictions. Since the flat

1 plate has no interior volume, the drag balance cannot be placed
2 within the plate. The cubic configuration of current drag
3 balances would cause considerable turbulence if placed between
4 the strut and the flat plate. If the drag balance were placed
5 above the strut, the drag forces exerted on the strut itself
6 would corrupt the drag reduction measurements. Additionally, The
7 size of current drag balances also prevents their placement
8 within an efficiently streamlined strut.
9

10 SUMMARY OF THE INVENTION

11 Accordingly, it is an object of the present invention to
12 provide a drag balance which does not create additional
13 turbulence when towed through a medium.

14 Another object of the present invention is to provide a drag
15 balance which can be mounted in close proximity to a flat plate
16 model.

17 Still another object of the present invention is to provide
18 a drag balance which can be incorporated into a streamlined strut
19 or towing post.

20 Other objects and advantages of the present invention will
21 become more obvious hereinafter in the specification and
22 drawings.

1 In accordance with the present invention, a drag balance is
2 provided within a hollow region of a streamlined strut. The drag
3 balance has a ground frame attached inside the strut and a
4 balance frame attached to the flat plate at the base of the
5 strut. The ground frame and balance frame are connected by
6 flexures which allow only a small amount of lateral movement of
7 the balance frame and model relative to the ground frame and
8 strut. Strain gages are attached to one or more of the flexures
9 to determine the amount of bending in the flexures and thus
10 obtain a measure of the drag forces exerted on the model. Since
11 the drag balance is mounted within the streamlined strut,
12 turbulence is minimized. The drag balance is also connected
13 directly to the model at the base of the strut such that drag
14 forces on the model can be accurately determined. Further, the
15 ground frame and balance frame can be easily shaped to conform
16 within an efficiently streamlined strut.

17 18 BRIEF DESCRIPTION OF THE DRAWINGS

19 A more complete understanding of the invention and many of
20 the attendant advantages thereto will be readily appreciated as
21 the same becomes better understood by reference to the following
22 detailed description when considered in conjunction with the
23 accompanying drawings wherein corresponding reference characters

1 indicate corresponding parts throughout the several views of the
2 drawings and wherein:

3 FIG. 1 is a schematic representation of a drag reduction
4 measurement test system using the drag balance of the present
5 invention;

6 FIG. 2 is a side view of the drag balance of the present
7 invention shown mounted within a cross section of a strut;

8 FIG. 3 is a cross sectional view taken along the line 3-3 of
9 FIG. 2 showing a top view of the drag balance of the present
10 invention mounted within a streamlined strut; and

11 FIG. 4 is an isometric view of the drag balance of the
12 present invention.

13 14 DESCRIPTION OF THE PREFERRED EMBODIMENT

15 Referring now to FIG. 1, there is shown a schematic
16 representation of a drag reduction measurement system 10. A tow
17 carriage 12 rides on rails 14 above the surface of medium 16. A
18 streamlined strut 18 is suspended from carriage 12 and extends
19 into medium 16. Strut 18 is hollow and has a drag balance, shown
20 by dashed outline 20, mounted within its lower portion. A flat
21 plate 22 is attached to the lower end of drag balance 20. As
22 carriage 12 moves along rails 14, strut 18 and plate 22 are moved

1 through medium 16. Drag forces exerted by medium 16 on plate 22
2 are transferred to and measured by drag balance 20.

3 Referring now to FIGs. 2 through 4, FIG. 2 shows a side
4 cross sectional view of the lower end of strut 18 with drag
5 balance 20 mounted therein. FIG. 3 shows a cross section taken
6 along line 3-3 of FIG. 2 and FIG. 4 shows an isometric view of
7 drag balance 20. Drag balance 20 consists of upper and lower
8 ground frames, 24a and 24b, and upper and lower balance frames,
9 26a and 26b. It can be seen that frames 24a, 24b, 26a and 26b
10 are shaped to generally conform to the streamline shape of strut
11 18. Two ground columns 28 are spaced along the longitudinal axis
12 X-X of drag balance 20 and are attached between upper and lower
13 ground frames, 24a and 24b, passing through balance bores 30 in
14 upper balance frame 26a. Similarly, two balance columns 32 are
15 attached between upper and lower balance frames, 26a and 26b,
16 along axis X-X and passing through ground bores 34 in lower
17 ground frame 24b. In the preferred embodiment shown, columns 28
18 and 32 are shown as round rods press fit into corresponding press
19 fit bores 36 in the frames. It will be readily understood that
20 the columns can be of generally any shape and can be affixed to
21 the frames in any suitable manner. The round shape of columns 28
22 and 32 shown allows for easy machining of bores 30 and 34. Press
23 fitting allows for easy assembly and disassembly of drag balance

20. A flexure plate 38 is attached at each end of the frames between upper ground frame 24a and upper balance frame 26a and between lower ground frame 24b and lower balance frame 26b. In the preferred embodiment shown, flexure plates 38 are fabricated of stainless steel feeler gage stock such that movement parallel to axis X-X is accommodated by bending of flexure plates 38 about an axis Y-Y, perpendicular to axis X-X, while the stiffness of flexure plates 38 about axis X-X prevents movement parallel to axis Y-Y. Flexure plates 38 are attached to the frames by cap screws 40 passing through flexure caps 42 and threading into the frames. It will also be readily understood that flexure plates 38 can be attached to frames 24a, 24b, 26a and 26b by any suitable means which would allow for their easy removal. Drag balance 20 is attached to flat plate 2 by means of bolts 44 passing through extensions 26c at either end of lower balance frame 26b. Drag balance 20 is secured inside strut 18 by means of securing screws 46 passing through strut 18 and threading into securing bores 48 on upper and lower ground frames 24a and 24b. In the embodiment shown, a total of four securing screws 46 are provided for each of ground frames 24a and 24b, two to each side of axis X-X. However, it will be understood that any suitable means of releasably securing ground frames 24a and 24b within strut 18 may be used. In order to maintain ground frames 24a and

1 24b and balance frames 26a and 26b steady during handling, two
2 lock pins 50 are provided, each pin extending through lower
3 ground frame 24b and into respective balance columns 32. Prior
4 to testing, pins 50 are removed to allow bending of flexure
5 plates 38. Lock pin bores 52 may be provided in strut 18,
6 aligned with pins 40, to facilitate removal of pins 50. In order
7 to measure the drag forces exerted on flat plate 22, strain gages
8 54 are provided on one or more of flexure plates 38 with data
9 leads 56 to a processor (not shown). Drag forces exerted on flat
10 plate 22 in the direction of axis X-X are transferred to bottom
11 balance frame 26b, which is directly attached to flat plate 22
12 and to top balance frame 26a through balance columns 32. The
13 drag forces result in the movement of balance frames 26a and 26b
14 relative to ground frames 24a and 24b, causing flexure plates 38
15 to bend. Strain gages 54 measure the amount of strain in flexure
16 plates 38 and hence the drag force causing the strain. Ground
17 frames 24a and 24b and balance frames 26a and 26b are also
18 provided with communication bores 58 for control wiring and the
19 like to be connected to flat plate 22 or other model being towed.

20 The drag balance thus described is compact and can be made
21 to easily fit within a streamlined strut. It provides a
22 convenient and accurate way to measure drag forces on a model
23 when a drag balance cannot be fitted within the model itself.

1 The drag balance is attached directly to the model for an
2 accurate measurement of the drag forces exerted on the model, yet
3 provides a streamlined shape to prevent undesired turbulence.

4 Although the present invention has been described relative
5 to a specific embodiment thereof, it is not so limited. The
6 exact materials and shapes described can be modified to suit the
7 testing conditions encountered. Various column shapes, including
8 square and I-shaped, may be used and the columns may be welded or
9 bolted to the frames. The material and thickness of the flexure
10 plates can be varied to suit the expected drag forces. Further,
11 the flexure plates may be welded to the frames at one end rather
12 than being screwed to the frames.

13 Thus, it will be understood that many additional changes in
14 the details, materials, steps and arrangement of parts, which
15 have been herein described and illustrated in order to explain
16 the nature of the invention, may be made by those skilled in the
17 art within the principle and scope of the invention,

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STRUT-MOUNTED DRAG BALANCE

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ABSTRACT OF THE DISCLOSURE

6 A drag balance for measuring drag forces on a model is
7 described which mounts within a hollow region of a streamlined
8 strut. The strut is attached to a carriage for towing the model
9 through a medium. The drag balance has a ground frame attached
10 inside the strut and a balance frame attached to the flat plate
11 at the base of the strut. The ground frame and balance frame are
12 connected by flexures which allow only a small amount of movement
13 of the balance frame and model relative to the ground frame and
14 strut in the towing direction. Strain gages are attached to one
15 or more of the flexures to determine the amount of bending in the
16 flexures and thus obtain a measure of the drag forces exerted on
17 the model. Mounting the drag balance within the streamlined
18 strut minimizes turbulence and allows the drag balance to be
19 connected directly to the model at the base of the strut such
20 that drag forces on the model are accurately determined.

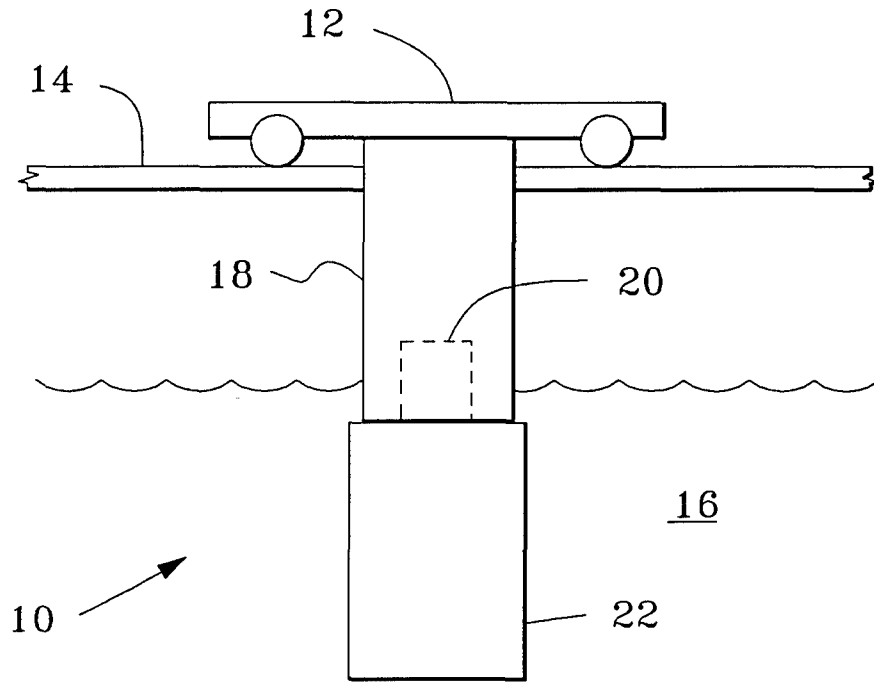


FIG. 1

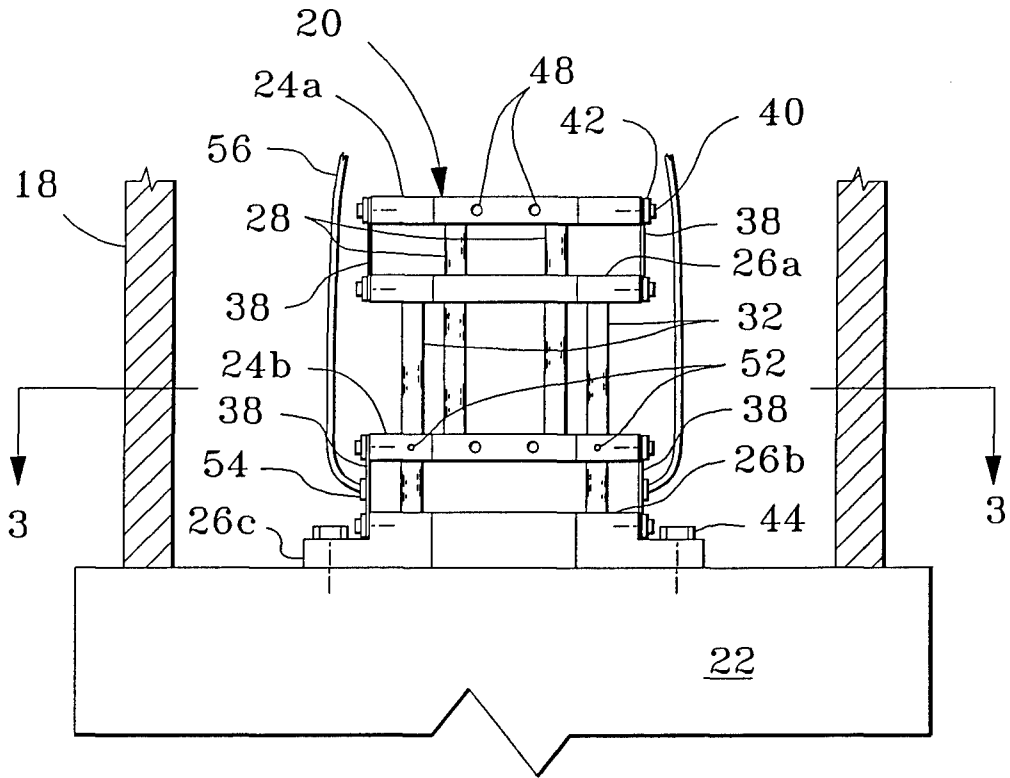


FIG. 2

