

The purpose of Volume 2 of "Hydraulic Design Criteria" is to prevent overcrowding of Volume 1 and to facilitate use of the design charts. To accomplish this purpose it will be necessary to divide Hydraulic Design Criteria from time to time as the number of charts increases. The revised tables of contents included with each new issue of Hydraulic Design Criteria will divide the charts in an appropriate manner.

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HYDRAULIC DESIGN CRITERIA

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) **GATES AND VALVES -300** (Continued)

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ARTIFICIAL **CHANNELS - 600**

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HYDRAULIC DESIGN CRITERIA

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Chart No.

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GATES AND VALVES - 300

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ARTIFICIAL CHANNELS - 600 (Continued)

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SHEW~S 310-1 To **310-1/2**

WAVE PRESSURES **ON** CREST **GATES**

1. A theory for the pressure resulting from a wave striking a verti**cal** wall was developed by Sainflou (1). The particular phenomenon is known as a "clapotis." The incident wave combines with the reflected wave to produce a wave height twice that of the incident wave. The theory is valid only for wave heights which do not exceed the still-water depth. The depth of water behind spillway crest gates is normally greater than the design wave height. Therefore, the theory can be used to estimate pressure distribution for the design of crest gates and for spillway stability analysis problems.

2. Application of the Sainflou wave pressure theory to crest gates **and** spillways.is illustrated on Hydraulic Design Chart **310-1.** The first equation is a parameter of the clapotis and indicates the effective change in mean water depth resulting from transition of the wave. The second equation indicates the change in bottom pressure. The clapotis results in pressure decrease **as** well as a pressure increase relative to the stillwater static pressure. Design problems are generally only concerned with the maximum pressure.

3. Overtopping of a gate **by** wses occurs when the clapotis rises above the gate. For this condition the maximum pressure distribution would be zero at the top of the gate and vary along a curve which would become asymptotic to the atraight-line distribution at the bottom of the spillway structure. As data are not available to establish the true pressure distribution, it may be assumed for design purposes that the portion of the pressure diagram above the top of the gate is ineffective and that the pressure distribution below the top of the gate is a straight line as indicated on Chart **310-1.**

4. The equations of the clapotis involve hyperbolic functions of the cosine and cotangent. Hydraulic Design Chart **310-1/1** presents graphical and tabulated values of these functions for depth-wave length ratios (D/λ) of 0.0 to 0.8.

5. Hydraulic Design Chart **310-1/2** is a sample computation illustrating use of the Sainflou theory for crest gate design and spillway stability analysis. A wave length, wave height, and approach depth of 125, 6, and 75 ft, respectively, have been assumed for the computation. The direction of approach is considered normal to the spillway.

(1) M. Sainflou, "Essay on vertical breakwaters," Annales des Ponts et Chaussees (July-August 1928), pp 5-48. Translated by C. R. Hatch for U. S. Army Engineer Division, Great Lakes, CE, Chicago, Ill. (No date.)

310-1 to ?10-1/2

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COMPUTATION SHEET PROJECT _______ John Doe Dam _______ SUBJECT _______ Crest Gates **JOB CW 804** COMPUTATION **Effects of Wave Pressure** COMPUTED BY RGC DATE 6/3/60 CHECKED BY MBB DATE 6/7/60

U. S. ARMY ENGINEER WATERWAYS EXPERIMENT STATION

GIVEN:

Gated spillway as shown Design wave length $(\lambda) = 125$ ft Design wave height $(H) = 6$ ft Still-water depth (D) = 75 ft

REQUIRED:

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1. Maximum pressure distribution on gate and spillway structure

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- 2. Maximum hydraulic load per ft of width of gate
- 3. Maximum hydraulic load per ft of width of structure

COMPUTE:

1. Pressure distribution

(a) Maximum effective depth with wave

$$
h_o = \frac{\pi H^2}{\lambda} \coth \frac{2\pi D}{\lambda} \quad \text{(Char 310.1)}
$$
\n
$$
\frac{D}{\lambda} = \frac{75}{125} = 0.6; \coth \frac{2\pi D}{\lambda} = 1.001 \quad \text{(Char 310-1/1)}
$$
\n
$$
h_o = \frac{3.14 \times 6^2}{125} \times 1.001 = 0.9 \text{ ft.}
$$

Effective depth = $D + h_o + H = 75.0 + 0.9 + 6.0 = 81.9$ ft.

(b) Maximum effective bottom pressure with wave

$$
a = \frac{H}{\cosh \frac{2\pi D}{\lambda}}
$$
 (Chart 310-1)

$$
\frac{D}{\lambda} \approx 0.6; \cosh \frac{2\pi D}{\lambda} = 21.7
$$
 (Chart 310-1/1)

$$
a = \frac{6}{21.7} = 0.3 \text{ ft.}
$$

Effective pressure = $D + a = 75.0 + 0.5 = 75.3$ ft.

PREPARED BY U. S. ARMY ENGINEER WATERWATS EXPERIMENT STATION, VICKSBURG, MISSISSIPPI

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- (c) Depth of gate overtopping 1.epth **. 81.9 - (75.0** -21.0 **+ 26.0)- 1.9 ft.**
- **(d)** Maximum pressure distribution graph

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2. Maximum hydraulic load per foot of width of gate (from **id** above)

Maximum pressure at top of gate $(P_1) = \frac{1.9}{81.9} \times 75.3 = 1.7$ ft **27.9** Maximum pressure at bottom of gate $(P_2) = \frac{27.9}{81.9} \times 75.3 = 25.7$ ft Maximum hydraulic load on gate $(R) = y \left(\frac{P_1 + P_2}{2} \right) \times$ gate height $y =$ specific weight of water = 62.4 lb/ft³ $R = 62.4 \left(\frac{1.7+25.7}{2}\right)26$ **-** 22,200 lb/ft of width

Note: For still-water level maximum gate pressure Ic 21 ft of water and maximum hydraulic load is 15 750 lb/tt of width.

3. Maximum hydraulic load per foot of width of (fiom **UA** above)

Maximum pressure at bottom of structure $(F_3) = 75.3$ ft

Maximum hydraulic load on structure $(R_h) = \gamma \left(\frac{P_1 + P_3}{2} \right) \times$ height of structure $=62.4 \left(\frac{1.7 + 75.3}{2} \right) 80$ **- 192,000** lb/ft of width

/t Note: Equivalent for still-water level is **175,000** lb/ft of width.

CREST **GATES** WAVE PRESSURE **SAMPLE COMPUTATION of** HYDRAULIC **DESIGN** CHART 310-1/2 **PAEPAREO** 6Y **U 3.** ARMY **ENGIkSIS WA9*?ESAYS CXPCRIMT.4 STATION. VICK60URG. MIS3iSSIPPI tsEE G;0**

(SHEETS 311-1 TO **311-5**

TAINTER GATES ON SPILLWAY **CRESTS**

DISCHARGE COEFFICIENTS

1. Discharge through a partially open tainter gate mounted on a spillway crest can be computed using the basic orifice equation:

 $Q = CA \sqrt{2gH}$

where,

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Q **=** discharge in cfs

C **=** discharge coefficient

 $A = area of orifice opening in ft²$

^H**=** head to the center of the orifice in ft.

The coefficient (C) in the above equation is primarily dependent upon the characteristics of the flow lines approaching and leaving the orifice. In turn, these flow lines are dependent upon the shape of the crest, the radius of the gate, and the location of the trunnion.

2. Discharge Coefficients. Chart **311-1** shows a plot of average discharge coefficients computed from model and prototype data for several crest shapes and tainter gate designs for nonsubmerged flow. Data shown are based principally on tests with three or more bays in operation. Dischatge coefficients for a single bay would be lower because of side contractions although data are not presently available to evaluate this factor. On this chart, the discharge coefficient (C) is plotted as a function of the angle (β) formed by the tangent to the gate lip and the tangent to the crest curve at the nearest point of the crest curve. The net gate opening is considered to be the shortest distance from the gate lip to the crest curve. The angle is a function of the major geometric factors affecting the flow lines of the orifice discharge. One suggested desi_on curve applies to tainter gates having gate seats located downstream from the crest axis. The other suggested design curve is based on tests with the gate seat located on the axis and indicates the effects of the masonry shape upstream from the crest axis.

3. Computation. Computation of discharge through a tainter gate mounted on a spillway crest is considerably complicated by the geometry involved in determining the net gate opening to be used in the orifice formula. The problem is simplified by fitting circular arcs to the crest

311-1 to **311-5**

curve used in the design of spillways. Chart 311-2 illustrates the necessary computations to obtain the net gate opening and the angle **1** described in paragraph 2, for tainter gates mounted on spillway crests shaped to $X^{1.85}$ = -2 $H_A^{0.85}$ Y. All factors are expressed in terms of the design head (H_A) . The method shown is applicable to other crest shapes. However, the accompanying design aids, Charts 311-3 and 311-4, apply only to standard crests.

4. To initiate the computations, Y_L/H_d values of the gate lip are assumed and corresponding values of X_L/H_d are computed (columns 1 to 6, Chart 311-2). These coordinates are then located on Chart 311-3 to determine the characteristics of a 'ubstitute arc. The substitute arc is then used to compute the net gate opening (columns 7 to 14). The point of intersection of the masonry line by the gate opening is determined by similar triangles (columns 14, 15, and 16). Design aid Chart 311-4 can be used to determine the Y_c/H_d coordinate of the gate opening and masonry line intersection (column 17), and also the slope of the masonry line (columns 18 and 19) which in turn combines with the slope of the gate lip tangent to form the angle **\$** (column 20). If graphical methods are preferred to analytical methods, a large-scale layout will enable the head, net gate opening, and the angle β to be scaled so that the discharge can be computed with fair accuracy.

5. Chart 311-5 is a sample computation of the steps involved in the development of a rating curve for a partially open tainter gate. The final computations are dimensional and are believed accurate to within $+$ 2 per cent, for gate opening-head ratios (G_{α}/H) less than 0.6.

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TAINTER GATES ON SPILLWAY CRESTS SAMPLE DISCHARGE COMPUTATIONS HYDRAULIC DESIGN CHART 311-5

* FROM HYDRAULIC DESIGN CHART 311-2
** FROM HYDRAULIC DESIGN CHART 311-1

CHART 311-5

DATE 8-27-54 SPILLWAY DISCHARGE COORDINATES FOR RATING CURVE (POOL VS DISCHARGE FOR VARIOUS GATE OPENINGS) SUBJECT_ **FORMULAS** CHECKED BY RRW Q = CG_oB_{129H} JOHN DOE DAM DATE 8-25-54 PROJECT_ DESIGN HEAD (H_a) = 37.0 FT
GATE WIDTH (B) = 42.0 FT GIVEN COMPUTED BY AAMS COMPUTATIONS JOB CW804

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SHEETS 311-6 AND 311-6/1

CREST **PRESSURES**

1. General. Pressures on standard spillways with partly open tainter gates are principally affected by the gate opening, gate geometry, and head on the gate. The effects of gate radius and trunnion elevation can be generally neglected within the limits of practical design.

2. Background. A laboratory study of the effects of gate seat location on pressures for standard shaped spillway crests (HDC **111-1** to 111-2/1) was made at WES¹ prior to 1948. A design head of 0.75 ft was used. The results of an extensive study by Lemos² of all geometric variables including gate seat locations upstream and downstream of the crest were published in 1965. A design head of 0.5 ft was used in this study. Comparable model³ and prototype⁴ data are also available.

3. Design Criteria. Dimensionless crest pressure profiles for small, medium, and large gate openings for the design head and 1.33 times the design head are given in **HDC** 311-6 and 311-6/1. The data are for gate seat locations of from $0.0H_d$ to $0.6H_d$ downstream of the crest. The study by Lemos² included gate seat locations from $-0.2H_d$ upstream to $0.6H_d$ downstream of the crest, gate radii of 1.0 and 1.25H_d, trunnion elevations of from 0.2 to $1.0H_d$ above the crest, and heads of 1.0 and $1.25 H_d$. Lemos' results indicate that the minor relative differences in gate radii, trunnion elevations, and gate openings of the experimental data shown on charts 311-6 and 311-6/1 should have negligible effect on crest pressures estimated from the charts. The Chief Joseph³ and Altus⁴ model curves were interpolated from observed data.

4. Application. The data given in the charts should be adequate for estimating crest pressures to be expected under normal design and operating conditions. When unusual design or operating conditions are encountered, the extensive work of Lemos can be used as a guide in estimating pressure conditions to be expected.

5. The data presented in charts 311-6 and 311-6/1 show that crest pressures resulting from normal design and operation practices are not controlling design factors. For partial gate openings the expected minimum crest pressures may range from about -0.1H_d for pools at design head to about -0.2 H_d for heads approximating 1.3 H_d . Gated spillways are presently being built with 50-ft design heads; so for an underdesigned crest, the minimum pressure to be expected with gate control would be about **-10** ft of water. This pressure would increase to -5 ft if design head was the maximum operating head. Pressures of these magnitudes should be free of cavitation. Periodic surges upstream of partially open tainter gates have been observed for certain combinations of head and gate width. Criteria for

> $311-6$ and $311-6/1$ Revised 7-71

surge prevention are given in ETL 1110-2-51.⁵

6. The pressure profiles in charts 331-6 and 311-6/1 can **be** used to estimate crest pressures for the design head for various gate openings and gate seat locations. The general absence of excessive negative pressures is noteworthy. Structural economy should no doubt have a strong influence on the selection of the gate seat location.

7. References.

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- (1) U. S. Army Engineer Waterways Experiment Station, CE, General Spillway Tests (CW 801). Unpublished data.
- (2) National Laboratory of Civil Engineering, Department of Hydraulics, Ministry of Public Works, Instability of the Boundary Layer - Its Effects Upon the Concept of Spillways of Dams, by F. **0.** Lemos. Proceedings 62/43, Lisbon, Portugal, **1965.** WES Translation No. 71-3 by Jan C. Van Tienhoven, August 1971.
- (3) U. S. Army Engineer Waterways Experiment Station, CE, Prototype Spillway Credt Pressures, Chief Joseph Dam, Columbia River, Washington. Miscellaneous Paper No. 2-266, Vicksburg, Miss., April 1958.
- (4) Rhone, T. J., "Problems concerning use of low head radial gates." Proceedings of the American Society of Civil Engineers, Journal of the Hydraulics Division, paper 1935, vol 85, No. HY2 (February 1959).
- **(5) U. S.** Army, Office, Chief of Engineers, Engineering and Design; Design Criteria for Tainter Gate Controlled Spillways. Engineer Technical Letter No. 1110-2-51, Washington, D. C., 22 August 1968.

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SHEET 312

VERTICAL **LIFT GATES ON** SPILLWAYS

DISCHARGE COEFFICIENTS

1. Purpose. Vertical lift gates have been used on high-overflowdam spillways. However, they are more commonly found on low-ogee-crest dams and navigation dams with low sills where reservoir pool control normally requires gate operation at partial openings. Hydraulic Design Chart 312 provides a method for computing discharge for partly opened, vertical lift gates.

2. Background. Discharge under high head, vertical lift gates can be computed using the standard orifice equation given in Sheets 311-1 to 311-5. The equation recommended by King^I for discharge through low head orifices involves the head to the three-halves power. For flow under a low head gate, this equation can be expressed as

$$
Q_{G} = C_{d1} \sqrt{2g} \text{ L} \left(H_{2}^{3/2} - H_{1}^{3/2} \right) \tag{1}
$$

where Q_G is the gate controlled discharge, C_{dl} the discharge coefficient, g the acceleration of gravity, L the gate width, and H_1 and H_2 are the heads on the gate lip and gate seat, respectively.

3. A recent U. S. Army Engineers Waterways Experiment Station² study of discharge data from four laboratory investigations³⁻⁶ failed to indicate correlation of discharge coefficients computed using equation 1 above or the equation given in Sheets 311-1 to 311-5. However, the concept of relating gate-controlled discharge to free discharge was developed in that study. The free discharge equation is

$$
Q = C_d \sqrt{2g} \text{ LH}^{3/2}
$$
 (2)

where H is the head on the crest. The relation of controlled to free discharge was obtained by dividing equation 1 by equation 2.

$$
\frac{Q_{G}}{Q} = \frac{C_{d1}}{C_{d}} \left(\frac{H_{2}^{3/2} - H_{1}^{3/2}}{H_{3}^{3/2}} \right)
$$
(3)

4. Analysis. The analysis of data taken from references 3 through 7 indicated reasonable correlation between free and controlled discharge. The results are shown in Chart 312. This study indicated that the relation C_{d1}/C_d varied slightly with the discharge ratio but could be assumed

H **³¹²** Revised 1-68 as unity. Data from studies^{6,7} with the gate seat located appreciably downstream from the crest showed good correlation with data for on-crest gate seat locations.

6. References.

- (1) King, H. W., Handbook of Hydraulics for the Solution of Hydraulic Proclems, revised by E. F. Brater, 4th ed. McGraw-Hill Book Co., Inc., New York, N. Y., 1954, pp 3-9.
- (2) U. S. Army Engineer Waterways Experiment Station, CE, Discharge Rating Curves for Vertical Lift Gates on Spillway Crests, by R. H. Multer. Miscellaneous Paper No. 2-606, Vicksburg, Miss., October 1963.
- (3) **TT 3.** nt of Reclamation, Hydraulic Model Studies of Falcon Dam,_ by A. S. Reinhart. Hydraulic Laboratory Report No. HYD-276, July 1950.
- (4) _, ydraultc M0d.l Studies of Gorge High Dam Spillway and Outlet Works, by W. E. Wagner. Hydraulic Laboratory Report No. HYD-403, September 1955.
- (5) Carnegie Institute of Technology, Laboratory Tests on Hydraulic Models of Bluestone Dam, New River, Hinton, W. Va. Final report, prepared for the U. S. Army Engineer District, Huntington, W. Va., February 1937.
- (6) Case School of Applied Science, A Report on Hydraulic Model Studies for the Spillway and Outlet Works of Mahoning Dam on Mahoning Creek, Near runxsutawney, Pa., by G. E. Barnes. Prepared for the U. S. Army Engineer District, Pittsburgh, Pa., May 1938.
- (7) U. S. Bureau of Reclamation, Hydraulic Model Studies of Flaming Gorge Dam Spillway and Outlet Works, ry T. J. Rhone. Hydraulic Laboratory Reporu No. HYD-531, May **1964.**

312 Revised 1-68

SHEET 320-1

CONTROL **GATES**

DISCHARGE COEFFICIENTS

1. General. The accompanying Hydraulic Design Chart **320-1** represents test data on the discharge coefficients applicable to partial openings of both slide and tractor gates. The basic orifice equation is expressed as follows:

$$
Q = C G_{\text{o}} B \sqrt{2gH'}
$$

The coefficient C is actually a contraction coefficient if the gate is located near the tunnel entrance and the entrance energy loss is neglected. When the gate is located near the conduit entrance the head (H') is measured from the reservoir water surface to the top of the vena contracta. However, when the gate is located a considerable distance downstream of the conduit entrance, H' should be measured from the energy gradient just upstream of the gate to the top of the vena contracta because of appreciable losses upstream of the gate. The evaluation of H' requires successive approximation in the analysis of test data. However, the determination of **H'** in preparation of a rating curve can be easily accomplished by referring to the chart for C .

2. Discharge Coefficients. Discharge coefficients for tractor and slide gates are sensitive to the shape of the gate lip. Also, coefficients for small gate openings are materially affected by leakage over and around the gate. Chart **320-1** presents discharge coefficients determined from tests on model and prototype structures having various gate clearances and lip shapes. The points plotted on the **100** per cent opening are not affected by the gate but rather by friction and other loss factors in the conduit. For this reason the curves are shown by dashed lines above **85** per cent gate opening.

3. Suggested Criteria. Model and prototype tests prove that the 45⁰ gate lip is hydraulically superior to other gate lip shapes. Therefore, the **450** gate lip has been recommended for high head structures. In the 1949 model tests leakage over the gate was reduced to a minimum. Correction of the Dorena Dam data for leakage results in a discharge coefficient curve that is in close agreement with the 1949 curve. The average of these two curves shown on Chart **320-1** is the suggested design curve. For small gate openings special allowances should be made by the designer for any expected excessive intake friction losses and gate leakage.

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4. Values from the suggested design curve are tabulated below for the convenience of the designer.

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SHEETS **320-2** TO 320-2/3

VERTICAL LIFT GATES

HYDRAULIC **AND** GRAVITY FORCES

1. Purpose. The purpose of HDC's **320-2** to 320-2/2, which apply to the hydraulic forces on vertical lift gates, is to make the results of **in**vestigations of such forces available in a convenient nondimensional form. These charts are equally applicable to tractor gates and slide gates.

' 2. Definition. HDC **320-2** is included to simplify the definition of the hydraulic forces involved. For purposes of discussing buoyancy, a gate may be assumed to be a rectangular parallelepiped with the vertical axis coincident with the direction of gravity. If the body is completely inclosed, the buoyant force in still water is equal to the difference between the total pressure on top (downthrust) and the total pressure on the bottom (upthrust). For such an inclosed vertical body, water pressure on the upstream face has no vertical component of pressure.

3. Some engineers use the expression, the "wet weight" of a gate. This is simply the dry weight in air minus the buoyant force. If the body is cellular or lacks an upstream skin plate, the wet weight differs from that of a completely inclosed body. The gate shown in **HDC** 320-2 is an inclosed body and is further considered to have no horizontal projections such as gate seals.

4. The unit pressure on top of the gate, or downthrust, is dependent on the head of water in the gate well or the pressure head in the bonnet. This head in turn depends on the relation of the pressure difference across the gap and the area of the upstream gap coupled to the pressure differences and area of the downstream gap. Actually, the flow across the top of the gate has a hydrodynamic effect; but, for the purpose of these charts, this effect is not considered important.

5. The hydrodynamic effect of water flowing past the bottom of the gate is substantial. A reduction of pressure on the bottom from the theoretical static head is generally called "downpull," which may be viewed either as a reduction in upthrust or a reduction in buoyancy. Downpull is dependent upon the geometry of the gate bottom. HDC's 320-2 to 320-2/3 are concerned principally with the 45-degree gate bottom, for which experimental data are presented.

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6. Vertical Stability. The gate well can be sucked completely dry of water with certain combinations of upstream and downstream gap areas between the gate and the roof of the conduit. If the upthrust then exceeds the weight of the gate, the entire body of the gate will be thrust

> 320-2 to 320-2/3 Revised **10-61** {

vertically upward. The experimental data on upthrust are of value in checking the design for such a possibility. However, discharge coefficients for the upstream and downstream gaps must be assumed to determine whether a gate opening exists that could cause a practically dry well.

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7. Upthrust. Dimensionless plots of unit upthrust on the sloping bottom of four 45-degree gate-bottom designs are shown in HDC 320-2/1. The data sources are listed in paragraph **11.** The data include both model and prototype pressure measurements. The Fort Randall gate has a downstream skin plate and downstream seals, and the 45-degree sloping gate bottom has an upstream- skin plate. The Pine Flat and Norfork gates have upstream skin plates and downstream seals.

8. The upthrust force was computed from observed pressure data on **the** sloping gate bottom. These data were plotted on the horizontal plane of projection of the gate bottom. Pressure contours in feet of water were drawn, integrated, and divided by the area of projection between the conduit walls to determine the upthrust per unit area of cross section. The plots of data indicate that the conduit width-average gate thickness ratio is a factor in the magnitude of upthrust per unit area. The average gate thickness includes the gate bottom seal.

9. Pressure per unit area on top of the gate can be determined from HDC 320-2/2. The Fort Randall Dam data shown in the chart are based on field and model measurements of gate-well water-surface elevations. The Pine Flat and Norfork Dam data result from field measurements of bonnet pressures at these structures. Details of clearances between the gates and the gate recesses are also shown. The area of the top of the gate to be used in computation of the downthrust should include the area of the gate within the gate slots, the area between the conduit walls and the area of the gate top seal.

10. Application. HDC 320-2/3 is a sample computation illustrating the use of \overline{HDC} 's $\overline{320-2}/1$ and $\overline{320-2}/2$ in the solution of a hydraulic and gravity force problem. In this computation the hydraulic force is based on the cross-sectional area of the gate between the conduit walls. In actual design, the effects of the top and bottom gate seals and the area of the gate within the gate slots should also be considered.

11. Data Sources.

- **(1)** U. S. Army Engineer Waterways Experiment Station, CE, Vibration, Pressure and Air-Demand Tests in Flood-Control Sluice, Pine Flat Dam, Kings River, California. Miscellaneous Paper No. 2-75, Vicksburg, Miss., February 1954, and subsequent unpublished test data.
- (2) **_,** Slide Gate Tests, Norfork Dam, North Fork River, Arkansas. Technical Memorandum No. 2-389, Vicksburg, Miss., July 1954.
- (3) , Vibration and Pressure-Cell Tests, Flood-Control Intake

320-2 to **320-2/3** Revised **10-61**

320-2 to **320-2/3** Revised **10-61**

Gates, Fort Randall Dam, Missouri River, South Dakota. Technical **Report No. 2-435, Vicksburg, Miss., June 1956**.

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U. S. ARMY **ENGINEER** WATERWAYS EXPERIMENT **STATION COMPUTATION SHEET**

GIVEN:

Gate **-** Pine Flat type **(HDC 320-2/1)** Height **(D) = 9.0** Width (B) **- 5.0** Average thickness (T) **=** 1.2 **ft** Upstream gate clearance = 0.4 in. Downstream **gate** clearance **- 1.5** in. Dry weight $(W) = 8$ tons Gate opening **(G.) 3.0** ft

DETERMINE:

1. Energy **head** above conduit invert (H) Gate opening **(Go)** percent

 $\frac{G_o}{D}$ × 100 **=** $\frac{3}{9}$ × 100 = 33.3 Velocity **of** jet **(V.) ^Q**¹²⁰⁰ **= =108.5** ft/sec

$$
\frac{120}{100} = \frac{120}{0.737 \times 3 \times 5} = 108.5 \text{ ft/sec}
$$

Velocity head of j et $(V, 2/2g)$

$$
\frac{V_1^2}{2g} = \frac{(108.5)^2}{64.4} = 182.8 \text{ ft}
$$
 4. Hoist load (P) (HDC 320-2)

Energy head above conduit invert

 $H = CG_0 + V_1^2/2g$ $=(0.737 \times 3) + (182.8) = 185.0$ ft $= 8-1.6 = 6.4$ tons

$$
\frac{U_f}{H} = 0.51
$$
 for G_o = 33.3 percent
 U_f = 0.51 (185.0) = 94.4 ft
 U_f = 0.51 (185.0) = 94.4 ft

Gate well water surface above conduit **be considered.**

invert (H_w) **H**

$$
\frac{W}{H} = 0.53
$$
 for G_o = 33.3 percent

$$
d_i = H_w - (D + G_o) :
$$

$$
= 80.0 \text{ ft}
$$

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$$
P = W + A (d_i - u_i) y
$$

$$
= 8 + (5 \times 1.2) (86.0 - 94.4) 0.0312
$$

- **2.** Unit upthrust (u_f) . **Repeat Computations for other gate openings** 2. Unit upthrust (u_f) is a stocking to develop gate hoist load curve.
	- For Pine Flat from HDC 320-2/1
 $\frac{u_f}{H} = 0.51$ for G_o = 33.3 percent **finally and the gate supplying from the friction between the gate and the gate** $\frac{u_f}{H}$ **= 0.51 for G_o = 33.3 percent for any pulles has not be**
- **3. Unit downthrust** 2. In actual problems the difference be tween the projected areas of the top and bottom of the gate including seals For Pine Flat from HDC 320-2/2 **and areas within the gate including seasure**

Hw = 0.53 (185.0) = **98.0** ft VERTICAL LIFT **GATES**

Unit downthrust **Matter Control CONTRACT AND GRAVITY FORCES = Hw - (D + G)** = **98.0 - (9 +3)** SAMPLE COMPUTATION

= 86.0 ft HYDRAULIC **DESIGN** CHART **320-2/'**

SHEET 320-3

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TAINTER **GATES** IN **CONDUITS**

DISCHARGE COEFFICIENTS

1. HDC 320-3 presents coefficient curves for tainter gates in conduits for use in the discharge equation:

 $Q = C G \frac{B \sqrt{2gH}}{B}$

The coefficient C is actually a contraction coefficient when the head H is measured from the energy gradient just upstream from the gate to the top of the vena contracta downstream.

2. The curves shown in **HDC 320-3** are based on an equation by R. von Mises* for the contraction coefficient for two-dimensional flow through slots. The solution of this equation requires successive approximation of the contraction coefficient. The computations were made on an electronic digital computer. The sketch shown in the chart is considered to be a half-section of the symmetrical slot condition investigated by Von Mises. The conduit invert represents the center line of his geometry and the roof one of the parallel approach boundaries. The tangent to the gate lip is assumed to be the sloping boundary from which the jet issues. The plotted data result from controlled tests on the Garrison bunnel model** in which leakage around or over the gate was negligible and discharge under the gate was carefully measured. The agreement between the curves and Garrison data indicates the applicability of the curves to tainter gates in conduits with straight inverts.

***** Mises, R. von, "Berechnung von Ausfluss **-** und ueberfallzahlen (Computation of coefficients of out-flow and overfall)," Zeitschrift des Vereines deutscher Ingenieure, Band 61, Nr. 22 (2 June 1917), p 473.

****** U. S. Army Engineer Waterways Experiment Station, CE, Outlet Works and Spillway for Garrison Dam, Missouri River, North Dakota, Technical Memorandum No. 2-431 (Vicksburg, Miss., March 1956).

1> 320-3 Revised **10-61**

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SHEETS 320-4 TO **320-7**

TAINTER GATES IN OPEN CHANNELS

DISCHARGE COEFFICIENTS

1. Free discharge through a partially open tainter gate in an open **PAINTER GATES IN OPE**

p channel can be computed using the equation:
 $\theta = C.C. G + B\sqrt{2}$

$$
Q = C_1 C_2 G_0 B \sqrt{2gh}
$$

The coefficient (C_1) depends on the vena contracta, the shape of which is a function of the gate opening (G_0) , gate radius (R) , trunnion height (a) , and upstream depth (h) for gate sills at streambed elevations. When the gate sill is above streambed elevation, the coefficient also depends upon sill height (P) and sill length (L).

2. Hydraulic Design Charts 320-4 to 320-6 present discharge coefficients (C_1) for tainter gates with sills at streambed elevation. The insert graphs on the charts indicate adjustment factors (C_2) for raised sill conditions. Charts are included for a/R ratios of 0.1, 0.5, and 0.9. Coefficients for other a/R values can be obtained by interpolation between the charts. The coefficient is plotted in terms of the h/R ratio for G_0/R values of 0.05 to 0.5. The effect of G_0/h is inherent in the solution and is indicated by the limit-use curve $G_0/\tilde{h} = 0.8$.

3. The basic curves on Charts 320-4 to 320-6 were prepared from tests reported by Toch (3), Metzler (2), and Gentilini (1). The method of plotting was developed by Tocn. Cross plots of the Toch, Metzler, and Gentilini data resulted in the interpolated curves. Good correlation of test results was obtained for the larger gate openings. Similar correlation was not obtained in all cases for the smaller gate openings. The Gentilini data for the smaller G_0/R ratios and their general correlation with Metzler's data resulted in the interpolated curves for G_0/R values of 0.05 and 0.1. The 0.2 curve is in close agreement with results reported by Toch. Interpolated coefficients from the C_1 curve indicate general agreement with experimental results to within **+3** per cent.

4. Charts 320-4 to 320-6 also apply to raised sill design problems when the adjustment factor curve shown on the auxiliary graph is considered. The C2 curve was developed from U. S. Army Corps of Engineers (4-7) studies and indicates the effects of the L/P ratio on the discharge coeffi-Experiment with experimental results to within ± 3 per cent.

4. Charts 320-4 to 320-6 also apply to raised sill design problems

when the adjustment factor curve shown on the auxiliary graph is consider

The C₂ curv data. Sufficient information is not available to determine the effects, if any, of the parameter P/R.

320-4 to 320-Y

5. Hydraulic Design Chart 320-7 is a sample computation sheet illustrating application of Charts 320-4 to 320-6.

- 6. References.
- (1) Gentilini, B., "Flow under inclined or radial sluice gates technical and experimental results." La Houille Blanche, vol 2 (1947), p 145. WES Translation No. 51-9 by Jan C. Van Tienhoven, November 1951.
- (2) Metzler, D. E., A Model Study of Tainter Gate Operation. State University of Iowa Master's Thesis, August 1948.
- (3) Toch, A., The Effect of a Lip Angle Upon Flow Under a Tainter Gate. State University of Iowa Master's Thesis, February 1952.
- (4) U. S. Army Engineer Waterways Experiment Station, CE, Model Study of the Spillway for New Lock and Dam No. **1,** St. Lucie Canal, Florida. Technical Memorandum No. 153-1, Vicksburg, Miss., June 1939.
- (5) , Spillway for New Cumberland Dam, Ohio River, West Virginia. Technical Memorandum No. 2-386, Vicksburg, Miss., July 1954.
- (6) , Stilling Basin for Warricr Dam, Warrior River, Alabama. Technical Report No. 2-485, Vicksburg, Miss., July 1958.
- (7) , Spillways and Stilling Basins, Jackson Dam, Tombigbee River, Alabama. Technical Report No. 2-531, Vicksburg, Miss., January 1960.

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U. S. ARMY **ENGINEER** WATERWAYS EXPERIMENT **STATION COMPUTATION SHEET**

<u> Standard</u>

WES 0-6O

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HYDRAULIC DESIGN CRITERIA **SHEETS** 320-8 **Al)** 320-8/1 TAINTER **GATES** IN **OPEN CHANNELS** DISCHARGE COEFFICIENTS **SUBMERGED** FLOW

1. Tainter gates on low sills at navigation dams frequently operate at tailwater elevations resulting in submerged flow conditions. The discharge under the gate is controlled by the difference in the upper and lower pool elevations, the degree of sill submergence by the tailwater, the gate opening, and, to a lesser extent, the stilling basin apron elevation. Hydraulic Design Charts 320-8 and 320-8/1 present discharge coefficient data for computing flows under tainter gates on low sills operating under submerged conditions.

2. Basic Data. The U. S. Army Engineer Waterways Experiment Station $(WES)^{\perp}$ has developed the following equation for computing flows under gates on low sills with tailwater elevations greater than gate sill elevation.

$$
Q = C_{\rm g} \ln_{\rm g} \sqrt{2gh} \tag{1}
$$

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***** where

i.

 $Q =$ discharge, cfs

- C_{g} = submerged flow discharge coefficient, a function of the sill submergence-gate opening ratio
- $L = bay width, ft$
- h = tailwater depth over sill, ft **s**
- $g =$ acceleration, gravitational, ft per sec²
- h = total head differential pool to tailwater, ft (including approach velocity head)

Equation 1 results in good correlation of experimental data when C_S is plotted as a function of the submergence-gate opening ratio (h_s/G_o) . The equation was developed by modifying the standard orifice equation as follows

$$
Q = CLG_0 \sqrt{2gh} \tag{2}
$$

320-8 and 320-8/1

$$
Q\left(\frac{G_O}{h_S}\right) = CLG_O \left(\frac{G_O}{h_S}\right) \sqrt{2gh}
$$

$$
Q = C_S LG_O \left(\frac{h_S}{G_O}\right) \sqrt{2gh}
$$

$$
Q = C_S L h_S \sqrt{2gh}
$$
(3)

where

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 $C_c = C(G_0/h_c)$ G_{\sim} = gate opening Q

3. Chart 320-8 presents the results of extensive model tests²,3,4,5 and limited prototype data.⁶ The plotted curves are based on careful measurements and are believed to be representative of the best available data. The model data and most of the prototype data were obtained with the gates adjacent to the test gate open the same amount as the test gate. The plotted curves indicate the effects of the relation of the elevation of the stilling basin apron to that of the gate sill. The portions of the curves having C_s values less than 0.1 are based on prototype gate openings of **1** ft or less and on model gate openings of about 0.05 ft. The experimental data are omitted from this chart in the interest of clarity. Chart 320-8/1 is included to illustrate the degree of data correlation resulting in the curves presented in Chart 320-8.

Application. The suggested design curve in Chart 320-8 should Ч. be ucefui for developing pool regulation curves for navigation dam spillways consisting of tainter gates on low sill3. The curves presented generally represent sill elevations about 5 ft above streambed and stilling basin apron elevations 3.5 to 31 ft below sill elevation. The Hannibal and Cannelton spillway sills are located about 15 and 19 ft above streembed, respectively. The height of the sill above the approach bed does not seem to be an important factor in submerged flow controlled by gates. However, the coefficient data presented include all the geometric effects of each structure as well as the effects of adjacent gate operation. The curve most applicable to spillway design conditions should be used for developed appricante to spirit and design

- 5. References.
- **(1)** U. S. Army Engineer Waterways Experiment Station, CE, Typical Spillway Structure for Central and Southern Florida Water-Control Project; Hydraulic Model Investigation, by J. L. Grace, Jr. Technical Report No. 2-633, Vicksburg, Miss., September 1963.

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320-8 and 320-8/1

- (2) , Spillway, Millers Ferry Lock and Dam, Alabama River, Alabama; Hydraulic Model Investigation, by G. A. Pickering. Technical Report No. 2-643, Vicksburg, Miss., February 1964.
- (3) ________, Spillway for Typical Low-Head Navigation Dam, Arkansas River, Arkansas; Hydraulic Model Investigation, by J. L. Grace, Jr. Technical Report No. 2-655, Vicksburg, Miss., September 1964.
- (4) , Spillway for Cannelton Locks and Dam, Ohio River, Kentucky and Indiana; Hydraulic Model Investigation, by G. A. Pickering and J. L. Grace, Jr. Technical Report No. 2-710, Vicksburg, Miss., December 1965.
- **(5)** , Spillway, Hannibal Locks and Dam, Ohio River, Ohio and West Virginia; Hydraulic Model Investigation. Technical Report No. 2-731, Vicksburg, Miss., June 1966.
- (6) Denzel, C. W., Submerged Tainter Gate Flow Calibration. 1965, U. S. Army Engineer District, St. Louis, Mo. (unpublished memorandum).

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HYDRAULIC DESIGN CRITERIA

SHEETS 330-1 **AND** 330-1/1

GATE VALVES

DISCHARGE CHARACTERISTICS

1. The discharge characteristics of a flow control valve may be expressed in terms of a loss coefficient for valves along a full-flowing pipeline, or in terms of a discharge coefficient for free flow from a valve located at the downstream end of a pipeline. Loss and discharge coefficients for gate valves are given on Hydraulic Design Charts 330-1 and 330-1/1, respectively.

2. Loss Coefficient. The loss of head caused by a valve occurs not only in the valve itself but also in the pipe as far downstream as the velocity distribution is distorted. Tests to determine this total Note that we have the mode of the signed on the signed of the security of the secu sizes of gate valves at the University of Wisconsin(1) and the Alden Hydraulic Laboratory.(2) The results of these tests on the larger sizes of valves are given on Chart 330-1 as loss coefficients in terms of the velocity head immediately upstream from the valve. Data are given for both a simple disk gate valve having a crescent-shaped water passage at partial openings and a ring-follower type of gate valve having a lensshaped water passage at partial openings. The scatter in the Wisconsin data is attributed to minor variations in the geometry of the different makes of valves tested.

3. Discharge Coefficients. Discharge coefficients for free flow from a gate valve at the downstream end of a pipeline have been determined by the Bureau of Reclamation(3) for several makes and sizes of simple disk gate valves. The results of these tests are given on Chart $330 - 1/1$ as discharge coefficients in terms of the total energy head immediately upstream from the valve. The scatter in these data is attributed to minor variations in geometry of the valves tested.

4. Application. The loss data given on Chart 330-1 are applicable to valves installed in full-flowing pipelines having no bends or other disturbances within several diameters upstream and downstream from the valve. The discharge coefficients on Chart 330-1/1 are for valves installed at the downstream end of several diameters of straight pipe and discharging into the atmosphere.

5. List of References.

Æ. or)

(1) Corps, C. I., and Ruble, R. o., Experiments on Loss of Head in Valves and Pipes of One-half to Twelve Inches Diameter. University of Wisconsin Engineering Experiment Station Bulletin, vol. IX, No. **1.** Madison, Wis., 1922.

330-1 and **330-1/1**

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- **(2) Hooper, L. J., Tests of 4-, 8-, and 16-Inch Series 600 Rising Stem Valves for the W-K-M Division of ACE Industries, Houston, Texas.**
Alden Hydraulic Laboratory, Worcester Polytechnic Institute, Worcester, Mass., Sept. 1949.
- **(3) U. S.** Bureau **of** Reclamation, Study of Gate Valves and Globe Valves as Flow Regulators for Irrigation Distribution Systems Under Heads **Up** to About **125** Feet of Water. Hydraulic Laboratory Report No. **Hyd-337,** Denver. **Colo., 13** Jan. **1956.**

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FROM 8-TO 12-INCH-DIAMETER GATE VALVES
AT DOWNCTREAM END OF CONDUIT OF SAME FREE FLOW FREE FLOW **DISCHARGE COEFFICIENTS** HYDRAULIC **DESIGN** CHART **330-I/I** WES **6-57**

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SHEETS 331-1 to **331-3**

BUT'TERFLY **VALVES**

DISCHARGE **AND** HYDRAULIC **TORQUE** CHARACTERISTICS

1. The dis-harge and torque characteristics of butterfly valves can be expressed in terms of discharge and torque coefficients as functions of the angle of rotation of the valve vane from opened position. The discharge coefficient is primarily a function of the orifice opening whereas the hydraulic torque coefficient depends upon the geometry of the valve vane. Thus, differences in torque coefficients are to be expected for various shaped vanes at the same opening. Although considerable data have been published(2), only data indicated as the original computations or curves of the investigators have been included in Design Charts 331-1 to **331-2/1.**

2. Discharge Coefficients. A modified form of the standard orifice equation has been used for computation of valve discharge. The area used in the equation is based on the nominal diameter of the valve because of difficulty in determining the actual areas of the orifice openings for partially opened valves. The discharge coefficient varies inversely with the angle of rotation of the valve from opened position. Two valve locations have been tested; one in which the valve is near the outflow end of the pipe, and the other in which the valve is well within a straight reach of pipe. Hydraulic Design Chart 331-1 presents discharge coefficients for valves located within the pipe. Chart 331-1/1 presents similar data for valves located near the end of the pipe. The material used in these charts is taken from the following investigators: McPherson(7), Dickey-Coplen(4), Gaden(5), Colleville(8), DeWitt(3), and Armanet(1). The Dickey-Coplen data are from air tests on a thin circular damper. The Armanet tests reflect the effects of convergence in the valve housing downstream from the vane pivot.

3. Torque Coefficients. Torque coefficient data are presented in Charts 331-2 and 2/1. The available information is limited. Chart 331-2 pertains to valves located within the pipe and Chart 331-2/1 applies to valves located near the end of the pipe. The Keller and Salzmann(6) data in Chart 331-2 were obtained from air tests. The DeWitt curve in Chart $331-2/1$ was computed from published prototype torque curves. The Gaden curves are based on carefully controlled laboratory tests which included measurement of and correction for pressure distribution on the downstream face of the valve vane. The Armanet curves reflect the effects of convergence in the valve body. The scarcity of torque coefficient data is indicative of the need for torque tests on butterfly valves of American manufacture.

331-1 to 331-3

4. Application. A sample computation for torque is given in Chart 331-3. Final computations should be based on the recommendations of the valve manufacturer at which time friction torque and seating torque data should be considered.

5. List of References.

- **(1)** Armanet, L., "Vannes-Papillon Des Turbines." Genissiat, Numero Hors Serie De La Houille Blanche, pp 199-219.
- (2) Cohn, S. D., "Performance analysis of butterfly valves." Instruments, vol 24, No. 8 (August 1951), p 880-884.
- (3) DeWitt, C., "Operating a 2^{L} -in. butterfly valve under a head of 223 ft." Engineering News-Record (18 September 1930), pp 460-462.
- (4) Dickey, P. **S.,** and Coplen, H. L., "A study of damper characteristics." Transactions, ASME, vol 64, No. 2 (February 1942).
- (5) Gaden, D., "Contribution to study of butterfly valves." Schweizerische Bauzeitung, vol III, Nos. 21, 22, and 23 (May 21 and 28 and June 4, 1938). Similar material by D. Gaden was also published in England in Water Power (December 1951 and January 1952).
- (6) Keller, C., and Salzmann, F., "Aerodynamic model tests on butterfly valves." Escher-Wyss News, vol IX, No. **1** (January-March 1936).
- (7) McPherson, M. B., Strausser, H. S., and Williams, J. C., Jr., "Butter- (fly valve flow characteristics." Proceedings, ASCE, paper 1167, vol 83, No. HYl (February 1957).
- (8) Voltmann, Henry, discussion of reference 7. Proceedings, ASCE, vol 83, No. HY4 (August 1957), PP 1348-48 and 49.

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AP - TOTAL **ENERGY HEAD AT UPSTREAM** HYDRAULIC **DESIGN** CHART **331-2/I SIDE** OF VALVE **IN LB/SQ** FT

WES 4-54

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U. S. ARMY **ENGINEER** WATERWAYS EXPERIMENT **STATION**

COMPUTATION SHEET

GIVEN:

Total available head (H_T) = 225 ft **PRESSURE** Valve diameter (D) = 4 ft Valve shape -Gaden-Disk **A** on Chart **331.1** Energy **lass** in system without valve $(H_L) = 0.3 V^2/2 g$

L **ASSUME:**

$$
V = \frac{Q}{A} = 48 \text{ ft per sec}
$$

 $H_L = 0.3 H_v = 10 H$

COMPUTE:

1. Head loss (H_L) in system without valve 2. Required valve loss (ΔH) for $Q = 600$ cfs $V = \frac{Q}{I} = 48$ ft per sec **for all the example of** $\Delta H = H_T - H_L - H_v = 225 - 10 - 35 = 180$ Discharge coefficient (C_O) $H_v = V^2/2g = 35 \text{ ft}$
Q = C_o D² $\sqrt{a}\sqrt{\Delta H}$ (Chart 331.1)

$$
C_Q = \frac{600}{16 \times \sqrt{32.2} \times \sqrt{180}} = 0.49
$$

Frmsuggested design curve on Chart **331.1,** valve opening (α) = 36^o for C_Q of 0.49.

3. Hydraulic torque (T) for $Q = 600$ cfs and $\alpha = 36^\circ$. From Chart **331.2,** torque coefficient **(CT)** for Gaden.Disk **A** valve open **360 - 0. 10.**

> $T = C_T D^3 \Delta P$ (Chart 331-2) Where $\Delta P = (H_1 - H_2)y = \Delta Hy$ $T = 0.10 \times 64 \times 180 \times 62.5 = 72,000$ ft-lb

Repeat comnputations for other assumed discharges to determine discharge and hydraulic torque curves.

BUTTERFLY **VALVES SAMPLE COMPUTATION DISCHARGE AND TORQUE**

HYDRAULIC DESIGN CHART 331-3 WE3 **s-56**

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HYDRAULIC DESIGN CRITERIA **SHEETS 332-1 AND 1/1** HOWELL-BUNGER **VALVES** DISCHARGE COEFFICIENTS

1. General. The Howell-Bunger valve is essentially a cylinder gate mounted with the axis horizontal. A conical end piece with its apex upstream is connected to the valve body by vanes. A movable external horizontal sleeve controls the discharge by varying the opening between the sleeve and the cone. The discharge is in the form of a diverging hollow conical jet. Diameters of valves range from 1.5 to 9 ft. Some valves have four vanes while others have six vanes. Separate discharge coefficient charts are presented for four- and six-vane valves.

2. Discharge Coefficients. Discharge coefficients for Howell-Bunger valves have been computed for various dimensional features of the valves. However, the discharge coefficients shown on Charts 332-1 and 1/1 are based on the area of the conduit immediately upstream from the valve. The basic equation used is shown on each chart. The computed coefficients are plotted against the dimensionless factor, sleeve travel divided by conduit diameter.

3. Experimental Data. Discharge coefficients for Chatuge, Nottely, Watauga, and Fontana Dams were computed from prototype data published by the Tennessee Valley Authority⁽¹⁾. Coefficients for Ross Dam are based on model data published by the Bureau of Reclamation⁽²⁾. Coefficients for Nimrod Dam result from discharge measurements made by the Little Rock District, CE. Coefficients for Narrows Dam result from model data obtained by the Waterways Experiment Station. The data presented on Charts 332-1 and 332-1/1 indicate discharge coefficients of 0.82 and 0.87 for full openings of the four- and six-vane valves, respectively.

- (1) R. A. Elder and **G.** B. Dougherty, "Hydraulic Characteristics of Howell-Bunger Valves and Their Associated Structures," TVA Report dated 1 Nov. 1950.
- (2) "Investigation of Hydraulic Properties of the Revised Howell-Bunger Valve, City of Seattle, Washington," Hydraulic Laboratory Report No. 168, Bureau of Reclamation, April 1945.

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HYMAULIC BESIGN CRITERIA

SHEET 340-1

FLAP **GATES**

HEAD LOSS COEFFICIENTS

1. Flap gate head losses can be determined **by** the equation:

 $\underline{v^{\epsilon}}$ $\overline{2g}$

where

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 H_I = head loss in ft of water K = head loss coefficient V **=** conduit velocity in ft per sec

2. Hydraulic Design Chart 340-1 presents head loss coefficients for submerged flap gates. The data result from tests **by** Nagler **(1)** on 18-in.-, 24-in.-, and 30-in.-diameter gates.

3. Modern flap gates are heavier but similar in design to those tested **by** Nagler. It is suggested that Chart 340-1 be used for design purposes for submerged flow conditions until additional data become available. Head loss coefficient data are not available for free discharge.

(1) **F. A. Nagler, "Hydraulic tests of Calco automatic drainage gates,"** The Transit, State University of Iowa, vol **27** (February **1923).**

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SHEET 534-1

LOCK **CULVERTS**

REVERSE TAINTER **VALVES**

LOSS COEFFICIENTS

1. The head loss across a lock culvert valve can be determined from the equation:

 $H_{L} = K_{V} V^{2}/2g$

where

 H_{T_1} = head loss across the valve in ft of water $K_v =$ valve loss coefficient \dot{V} = mean culvert velocity in ft/sec $g =$ acceleration of gravity in ft/sec².

2. Hydraulic Design Chart 534-1 shows valve loss coefficients vs the ratio of the area of the valve opening to the area of the culvert for reverse tainter valves. The Weisbach curve (1) is based on data for a vertical gate in a rectangular conduit. The data shown were computed from model and prototype tests. A complete list of data sources is given in paragraph 3. The graph is similar to plate 6 of Engineer Manual 1110-2-1604. However, experimental data are plotted on Chart 534-1, to emphasize the excellent agreement of various test results.

3. Data Sources.

- **(1)** Weisbach. "Hydraulics and Its Application" by A. H. Gibson, D. Van Nostrand Co., Inc., New York, N. Y., 4th ed., 1930, p 249.
- (2) St. Anthony Falls Lower Lock Models l and 7. Unpublished data computed by U. S. Army Engineer District, St. Paul, Minnesota, under CW 820, December 1953.
- (3) McNary Lock Model, Test **1,** Run 1-C. Unpublished data computed by U. S. Army Engineer District, St. Paul, Minnesota, under CW 820, December 1953.
- (4) McNary Lock Prototype, Run 13-3. Report on Model-Prototype Conformity-McNary Dam Navigation Lock, 1955 Tests. U. S. Army Engineer District, Walla Walla, Washington, March 1959.

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(5) McNary Lock Prototype, **Run 9.** Unpublished **data computed by U. S. Army** Engineer Waterways Experiment Station, Vicksburg, Miss., from November
1957 tests.

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(6) Dalles Lock Model. Report on Model-Prototype Conformity-McNary Dam Navigation Lock, **1955** Tests. **U. S.** Army Engineer District, Walla Walla, Washington, March **1959.**

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HYDRAULIC DESIGN CRITERIA

SHEETS 534-2 **AND** 534-2/1

LOCK CULVERTS

MINIMUM BEND PRESSURE

RECTANGULAR SECTION

1. Laboratory flow studies have shown that, for a rectangular conduit section, the minimum pressure in circular bends of 90 to 300 deg occurs on the inside of the bend 45 deg from the point of curvature. Experimental turbulent flow pressure data, at this location, closely approximate values computed for two-dimensional potential flow. McPherson and Strausser¹ have suggested an analytical procedure for determining the magnitude of the minimum pressure in a circular bend of rectangular section.

2. Theory. The minimum bend pressure head can be computed from the equation

$$
C_p = \frac{H - H_i}{\frac{v^2}{2g}}
$$
 (1)

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where

 C_p = pressure-drop parameter

- $H =$ average pressure head, in ft, at the 45-deg point computed as a straight-line extensiun of the upstream pressure gradient
- **Hi =** minimum pressure head, in ft, at the 45-deg point on inside of bend
	- $V =$ average culvert velocity in ft per sec
	- $g =$ acceleration, gravitational, in ft per sec²

Equation 1 is similar to the bend coefficient equation developed by Lansford (reference 4, Sheet 228-3). Based on equation 3 of reference **1,** it can also be shown that

$$
C_p = \left[\frac{2}{\left(\frac{R}{C} - 1\right) \ln\left(\frac{\frac{R}{C} + 1}{\frac{R}{C} - 1}\right)} \right] - 1 \tag{2}
$$

534-2 and 534-2/1 Revised 1-68

where

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 $R =$ center-line radius of the bend

 $C = one-half the culvert width$

3. Application. Hydraulic Design Chart 534-2 shows the relation between the theoretical pressure-drop parameter and ratio of the radius of curvature to one-half the conduit dimension in the direction concerned. Values of C_p computed from experimental results reported by Silberman² and Yarnell and Woodward³ are also shown. These data indicate the effects of Reynolds numbers between 6.7×10^{4} and 8.2×10^{5} . Points computed from data summarized by McPherson and Strausser1 from tests by Addison, 4 Lell,⁵ Wattendorf,⁶ and Nippert⁷ and on the Waynesboro and Mt. Alto model studies at Lehigh University are included on the chart. The indicated Reynolds number is about **103** to **106.** The chart is considered applicable to bends of 45 to 300 deg.

4. Cavitation occurs when the instantaneous pressure at any point in a flowing liquid drops to the vapor pressure. Vapor pressure varies with temperature of the liquid (see Sheet 000-2). Since turbulence in flow causes pressure fluctuations, an estimate should be made of the maximum expected fluctuation from the minimum computed bend pressure. The sum of the estimated pressure fluctuation, the vapor pressure, and a few feet of water for a margin of safety should be computed. The local barometric pressure (see Chart 000-2) should be subtracted from this total to obtain the minimum permissible bend pressure. This pressure can then be used to determine the necessary average conduit pressure or the permissible average conduit velocity to prevent cavitation. Cavitation damage has been found where the average pressure is relatively high but violent negative pulsations reach cavitation pressures. Such criteria as indicated here should therefore be used conservatively.

5. Chart 534-2/1 is a sample computation showing the application of Chart 534-2 to the minimum bend pressure problem. Computations to indicate the minimum permissible average conduit pressure and the maximum permissible average conduit velocity to prevent cavitation are included. Chart $534-2$ can also be used for the design of bends in rectangular sluices and siphons and in circular conduits. Its application to the latter is shown in Chart $228-3$.

6. References.

- **(1)** McPherson, M. B., and Strausser, H. S., "Minimum pressures in rectangular bends." Proceedings, ASCE, vol 81, Separate Paper No. 747 (July 1955); vol 82, Separate Paper No. 1092 (October 1956), p 9, Closure.
- (2) Silberman, E., The Nature of Flow in an Elbow. Project Report No. 5, St. Anthony Falls Hydraulic Laboratory, University of Minnesota, Minneapolis, prepared for David Taylor Model Basin, December 1947.

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- (3) U. S. Department of Agriculture, Flo.. of Water Around 180-Degree Bends, by D. L. Yarnell, and S. M. Woodward. Technical Bulletin No. 526, Washington, D. **C.,** October 1936.
- (4) Addison, H., "The use of bends as flow meters." Engineering, vol 145 (4 March 1938), pp 227-229 (25 March 1938), p 324.
- (5) Lell, J., "Contribution to the Knowledge of Secondary Currents in Curved Channels (Beitrag zur Kenntnis der 3ekundärströmungen in gekrümmten Kanälen)." Dissertation, R. Oldenbourg, Muchen, 1913. Also Zeitschrift für das gesamte Turbinenwesen, Heft 11, July 1914, pp 129-135, 293-298, 313-317, and 325-330.
- (6) Wattendorf, F. L., "A study of the effects of curvature on fully developed turbulent flow." Proceedings, Royal Society of London, Series A, vol 148 (February 1935), pp 565-598.
- (7) Nippert, H., "Uber den Strbmungsverlust in gekrimmten Kanblen." VDI, Forschungsarbeiten, Heft 320, Berlin (1929).

534-2 and $534-2/1$ Revised 1-68

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COMPUTATION SHEET

Vmax **=** maximum **permissible** average conduit velocity (in **fps)** to prevent cavitation (H **= 10 ft).**

COMPUTE:

1. R/c = 10/5 = 2
 2. C_n = 2.30 for R/c = 2 (Chart 534-2)
 2. C_n = 2.30 for R/c = 2 (Chart 534-2)
 2. C_n = 2.30 for R/c = 2 (Chart 534-2) λ **bead (H_{min}) to prevent cavitation (V = 20 fps).**

minimum bend pressure (H₁)
\n
$$
\frac{H - H_1}{V^2/2g} = C_p
$$
\n
$$
\frac{H_{min} - H_{1,min}}{V^2/2g} = C_p
$$
\n
$$
\frac{H_{min} - (-17.8)}{20^2/64.4} = 2.30
$$

$$
H_{\text{min}} = 2.3 (400/64.4) - 17.8 = 14.3 - 17.8 = -3.5 \text{ ft}
$$

H₁ = -4.3 ft 6. Maximum permissible average conduit velocity 4. Minimum permissible bend **pressure** head (Vmox) to prevent ca",tation (conditions of **step** ⁴ **(Hi min)** and H **= 10 ft).**

$$
\frac{H - H_{1 \text{ min}}}{\sqrt{2} \text{ max}/2g} = C_p
$$
\n
$$
= 0.0 \text{ ft}
$$
\n
$$
= 0.4 \text{ ft (Sheet 000-2)}
$$
\n
$$
\frac{10 - (-17.8)}{\sqrt{2} \text{ max}/64.4} = 2.3
$$
\n
$$
\frac{10 - (-17.8)}{\sqrt{2} \text{ max}} = 2.3
$$
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$$
\frac{10 - (-17.8)}{\sqrt{2} \text{ max}} = 2.3
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$$
\frac{10 - (-17.8)}{\sqrt{2} \text{ max}} = 2.3
$$
\n
$$
\frac{10 - (-17.8)}{\sqrt{2} \text{ max}} = 2.3
$$

Note: Since HI > Hi mi cavitation should not occur. **However, this is not** adequate to use as posi- LOCK **CULVERTS tive criterion since the values used for items RECTANGULAR SECTION** of the do-eigner. **MINIMUM BEND PRESSURE SAMPLE** COMPUTATION HYDRAULIC **OESIGN** CHART 534-2/I **PREPARED BY U. 5. ARWY ENGINEER BATERBAYS EXPERIMENT STATION VICKSBURG MISSISSIPPI**

$$
\begin{array}{c}\n\bullet \\
\bullet \\
\bullet \\
\bullet\n\end{array}
$$

3. Minimum bend pressure (H₁)

a. Estimated pressure
head fluctuation = 10.0 ft h ead fluctuation **b. Vapor pressure head**

for margin of **safety = 5.0** ft

d. Local barometric **pressure head = 33.2 ft 2.3** 2. ⁷

e. Minimum **permissible bend pressure** head $(H_{\text{limit}}) = 15.4 - 33.2 = -17.8 \text{ ft}$

4a and 4c ore dependent upon the judgement

c. Pressure allowance

(Chart 000.2)

10 - Hⁱ = 2.30

of water at 50 F *=* **0.4** ft **(Sheet 000-2) 10** - **(-17.8) - 2.3**

HYDRAULIC DESIGN CRITERIA

SHEETS **610-i** to **610-7**

TRAPEZOIDAL **CHANNELS**

1. Hydraulic Design Charts **610-i** to **610-7** are design aids for reducing the computation effort in the design of trapezoidal channels having various side slopes from **1** to **1** to **3** to **1** with uniform subcritical or supercritical flow. It is expected that the charts will be of value in preliminary design work where different channel sizes, roughness values, and slopes are to be investigated. Certain features of the charts were based on graphs prepared **by** the Los Angeles District, **CE.** Charts **610-1** to **610-7** can be used to interpolate values for intermediate side slopes.

2. Basic Equations. Manning's formula for open channel flow,

$$
Q = \frac{1.486 \text{ A s}^{1/2} \text{ R}^{2/3}}{n}
$$

can be separated into a factor, involving slope and friction

$$
c_n = \frac{1.486 \text{ s}^{1/2}}{n}
$$

and a geometric factor involviug area and hydraulic radius

$$
c_{k} = AR^{2/3}
$$

Chart $610-1$ and $-1/1$ show values of the factor, C_n , for slopes of 0.0001 to 1.0 and n values of 0.010 to 0.035. Charts $610-2$ to $-4/1-1$ show values of the geometric factor, C_k , for base widths of 0 to 600 ft and depths of 2 to **30** ft. Chart3 610-5 to **-7** show values of critical depth divided by the base width for discharges of $1,000$ to 200,000 cfs and base widths of 4 to 600 ft.

3. Application. Preliminary design of trapezoidal channels for subcritical or supercritical flow is readily determined by use of the charts in the following manner:

- a. With given values of n and S, C_n can be obtained from charts **610-1** and **-1/1.** n
- **b.** Since $Q = C_n C_k$ the required value of C_k can be obtained by dividing the design Q by C_n .

610-i to 610-7 Revised 5-59

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- c. With the required **Ck** value, suitable channel dimensions can be selected from charts $610-2$ to $-4/1-1$.
- d. Charts 610-5 to 610-7 can be used to determine the relation of design depth to critical depth.

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ya sana ya mwana wa 1972 ya 2012 wa 教育学部のように、 4 65 60 55 50 45 40 VALUES OF C_K (THOUSANDS) 35 30 25 12 20 15 لملقل 2ام ن 10 $d = 10$ $d = 8$ 5 $rac{d=6}{d=4}$ $\circ \frac{1}{\circ}$ BO 100 120 20 40 $\overline{140}$ $\overline{160}$ 180 60 200 $C_K = AR^{2/3}$ WHERE A = AREA
R = HYDRAULIC RADIUS ž, TRAPEZOIDAL CHANNELS C_K VS BASE WIDTH SIDE SLOPE I TO I $\frac{1}{\epsilon}$ HYDRAULIC DESIGN CHART 610-2 ્ર WES 2-54 大学

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3.000 100 0.50 0300 $\frac{2}{3}$ CRITICAL DEPTH $0¹⁰$ 0050 0030 0010 0005 0003 DISCHARGE IN 1000 CFS 30 40 50 100 200 **BASIC FORMULA:** $Q = D_c^{3/2} \sqrt{\frac{(b \cdot Z D_c)^3}{b \cdot 2Z D_c}} \times g$ TRAPEZOIDAL CHANNELS T CRITICAL DEPTH CURVES $Z = \frac{e}{d}$ d SIDE SLOPE I TO I \overline{b} ⊶⊦⊸- e-⊶ HYDRAULIC DESIGN CHART 610-5 WES $2 - 54$

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HYDRAULIC DESIGN CRITERIA

i **SHEETS 610-8 To 610-9/1-1**

OPEN **CHANNEL FLOW**

RECTAIULAR SECTIONS

1. Hydraulic Design Charts **610-8** to 610-9/1-1 are aids for reducing the computation effort in the design **3f** rectangular channels. These charts are useful also in the backwater computations presented on Chart 010-2.

2. Basic Equations. Chart **610-8 shows** plots of normal depth **(yo)** with respect to discharge per foot of width **(q)** for wide rectangular sections where the side wall effect may be neglected. Normal depth curves ar- shown for Manning's n of **0.011** and **0.013** and for slopes of **0.01** to **0.50.** The roughness and slopes values are those comonly used in the design of spillway chutes. The curves are computed from a variation of the Manning formula for open channel **flow.**

$$
q = cy_0^{5/3}
$$

where

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$$
c = \frac{1.486 \text{ s}^{1/2}}{n}
$$

Critical depth (y_c) with respect to **q** is also plotted on this chart. Critical depth in rectangular channels is a function of unit discharge only

$$
y_c = \sqrt[3]{\frac{q^2}{g}}
$$

3. Charts **610-9** through **610-9/1-1** in conjunction with Charts **610-1** and $-1/1$ can be used to determine normal depths (y_0) for any rectangular channel. These charts are similar to Charts **610-2** to 610-4/1-1 and were developed in the manner described in paragraph 2 of Sheets **610-1** to **610-7.**

4. Application. Preliminary design of rectangular channels for uniform subcritical or supercritical flows is readily determined **by** use of the charts in the following manner:

> a. Two-dimensional flow. For wide channels, **yo** and yc can be obtained directly from Chart **610-8** for given values of n , **S** ,and **q**

> > **610-8** to 610-9/1-1 Revised **5/59**

b. Three-dimensional flow. For all channels, Charts **610-9** through 610-9/1-1 can be used in the manner described in paragraphs 3a, b, and c, Sheets **610-1** to 610-7. Critical depth can be obtained from Chart 610-8.

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c. Normal depth for three-dimensional flow can also be computed from Chart 610-8 by use of the following table:

where

- $b = channel width in ft$
- **d2 =** two-dimensional flow depth in ft
- d_3 = three-dimensional flow depth in ft.

610-8 to 610-9/1-i Revised 5/59

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HYDRAULIC DESIGN CRITERIA

SHEETS 623 TO 624-1

SUBCRITICAL OPEN CHANNEL FLOW

DROP STRUCTURES

1. Purpose. A channel invert slope can vary from a maximum defined by a line connecting the crests of two drop structures to a minimum fixed by the elevation of the end sill of the upstream, structire, the elevation of the crest of the downstream structure, and the distance between the two structures. The minimum siope should be that which results in stable channel conditions.

2. Hydraulic Design Charts (HDC's) 623 to 624-1 present design criteria for drop structures in subcritical flow used to prevent channel degradation. The criteria shown in HDC 623 are recommended for drcns where the unit discharge is large relative to the drop height. The design criteria shown in HDC 624 and $624/1$ are recommended for drop structures where both the unit discharge and drop height are large and where optimum energy dissipation is required to reduce downstream erosion. In most cases economy of construction is the deciding factor.

3. Background. The accepted relation between the height of drop h (difference in elevation between the crest and the end sill of the drop structure), critical depth d_c at the drop, and the required stilling basin length L_B is attributed to Etcheverry^L and defined by the equation

$$
L_B = C_L \sqrt{hd_c} \tag{1}
$$

where C_r is an empirical uptor length coefficient. Studies by Morris and Johnson² resulted in design of the CIT (California Institute of Technology) structure restricted to h/d_c ratios greater than 1.0. Subsequent studies by Vanon₃ and Pollak³ included ratios as low as 0.3. While initial research efforts were directed toward erosion control in gullies, subsequent application has been mostly in alluvial streams.

4. Donnelly and Blaisdell⁴ investigated drop structures having h/d_c ratios from 1 to 15 and developed the SAF drop structure for primary use in the control of erosion in gullies. The major difference in CIT and SAF structures is the difference in tailwater depths, i.e. shallow and deep, respectively.

5. CIT-Type Drop Structures. Extensive WES tests⁵ on the CITtype structure resulted in the design criteria given in HDC 623. The Vanoni and Pollak results appear to correlate well with the WES tests. WES tests showed that optimum structure performance is obtained if the

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structure is designed to have a tailwater-critical depth ratio between 1.25 and 1.67. This results in a strong ground roller, a confined, strong and stable surface roller, and a depressed secondary roller downstream. Curved, upstream abutment walls are recommended for narrow channels to help prevent concentration of the flow. For wide channels with flow width \geq 20 times the depth, rectangular abutments are satisfactory. Stilling basin training walls should be sufficiently high to prevent the tailwater returning over the walls into the stilling basin. Wing walls at the end of the basin are not recommended. The channel edge should be recessed as indicated in HDC 623.

 $6.$ SAF-Type Drop Structures. The SAF-type drop structure^{4,6} (HDC's 624 and 624-1) is recommended for designs having large unit discharges and drop heights. The basic layout is shown in HDC 624. The primary controlling parameter in this design is the location at which the upper nappe of the falling jet impinges on the stilling basin floor. This is a function of the total fall of the jet and the depth of the tailwater. Dimensionless curves for determining the impact location of the upper nappe on the basin floor are shown in HDC 624-1.

7. The dimensions of the stilling basin are computed from the following equations.

$$
L_B = X_a + X_b + X_c
$$
 (2)

where L_B equals basin length. HDC 624 graphically defines the distance X_{a} , X_{b} , and X_{c} . Numerical values of X_{b} and X_{c} are obtained from the following equations:

$$
x_{\rm p} = 0.8d_{\rm c} \tag{3}
$$

$$
X_c = 1.75d_c \tag{4}
$$

Substituting equations 3 and 4 into equation 1 results in

$$
L_B = X_a + 2.55d_c
$$
 (5)

with d_c as defined in paragraph 3 and as shown in HDC's 623 and 624. Laboratory tests⁴ have resulted in the following recommendations for baffle pier and end sill heights.

$$
Baffle pier height = 0.8d \tag{6}
$$

End sill height
$$
h' = 0.4d
$$
 (7)

These tests also showed that optimum basin performance occurs when the baffle pier width and spacing effect a 50 to 60 percent reduction in flow width and the minimum tailwater depth is not less than 2.15d_c.

623 to 624-1

8. Design Discharge. Design discharge for the drop structure should be computed using the equation

 $Q = C H^{3/2}$ (8)

where

第一章

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 $Q =$ design discharge, cfs C = discharge coefficient = $3.0*$ $L =$ length of the drop structure crest, ft $H = energy$ head on the crest, ft

The length L of the weir should effect optimum use of channel cross section upstream. A trial-and-error procedure should be used to balance the crest height and width with the channel cross section.

9. Riprap Protection. Riprap protection should be provided immediately upstream and downstream of each structure. It is recommended that design criteria given in **HDC** 712-1 be used to meet stilling requirements and that given in EM 1110-2-1601 (reference 7) for upstream protection.

10. References.

- **(1)** Etcheverry, B. **A.,** Irrigation Practice and Engineering. 1st ed., Chapter VII, McGraw-Hill Book Company, New York, N. Y., 1916.
- (2) Morris, B. T. and Johnson, D. C., "Hydraulic design of drop structures for gully control." Transactions, American Society of Civil Engineers, vol 108 (1943), pp 887-940.
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- (4) Donnelly, C. A. and Blaisdell, F. W., Straight Drop Spillwa Stilling Basin. Technical Paper No. **15,** Series B, St. Anthony Falls Hydraulic Laboratory, University of Minnesota, Minneapolis, Minn., November 1954.
- (5) U. S. Army Engineer Waterways Experiment Station, CE, Drop Structure for Gering Valley Project, Scottsbluff County, Nebraska, Hydraulic Model Investigation, by T. E. Murphy. Technical Report No. 2-760, Vicksburg, Miss., February 1967.
- (6) U. S. Department of Agriculture, Soil Conservation Service, Engineer-Handbook, Drop Spillways. Section **11,** Type C, Washington, D. C., p **5-11.**

Reduced for submergence effects when applicable.
(7) U. S. Army, Office, Chief of Engineers, Engineering and **Design;** Hydraulic Design of Flood Control Channels. Engineer Manual EM **1110-2-1601,** Washington, **D. C., 1** July **1970.**

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THE CREST, NEGATIVE WHEN TAILWATER SURFACE IS ABOVE **THE CREST, NEGATIVE WHEN TAILWATER CONDUMNEL FLOW**

SAF- TYPE DROP STRUCTURE REDRAWN FROM FIG. SHYDRAULIC *2,* **REFERENCE 4 JET IMPACT DESIGN LOCATION CHART 024-I**

HYDRAULIC DESIGN CHART 624-1

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HYDRAULIC DESIGN CRITERIA

SHEETS 631 TO 631-2

OPEN CHANNEL FLOW

RESISTANCE COEFFICIENTS

1. General. Because of its simplicity, the Manning equation has been used extensively in the United States in the evaluation of resistance losses in open channel flow. A comprehensive summary of the use of this equation in channel design is given in reference 1. Flow data and Manning's n's for 50 natural streams, together with color photographs of the channels, have also been published.² The Chezy equation¹ includes a resistance coefficient term that is applicable to all flow conditions. Hydraulic Design Chart 631 presents a general resistance diagram relating Chezy's C , Reynolds number, and relative roughness. The chart is useful in open channel flow problems.

2. Laboratory and field investigations have shown that the resistance coefficient varies with Reynolds numbers as well as with boundary surface roughness. Keulegan³ has demonstrated that the Von Karman-Prandtl smooth and rough pipe resistance equations based on the Nikuradse test data can be applied to open channel flow with only minor adjustments in the equation constants. A recent ASCE progress report⁴ recommends a Moody-type diagram for use in open channel flow, especially for flows in which the viscous effects are important.

3. Chezy Equation. The Chezy equation is

 $V = C \sqrt{RS}$

where

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 V = mean channel velocity, ft per sec

C **=** Chezy resistance coefficient which is a function of Reynolds number and relative roughness of channel

 $R = hydraulic radius of channel, ft$

S = slope of energy gradient

4. Resistance Coefficient Relations. The Darcy resistance coefficient f (see Hydraulic Design Chart 224-1) is defined as

$$
f = \frac{8RSg}{v^2}
$$

where $g = acceleration$ of gravity.

The relation between C and f is

$$
C = \sqrt{\frac{8g}{f}}
$$

Similarly, the relation of C and n can be shown to be

$$
C = \frac{1.486R^{1/6}}{n}
$$

5. Effects of Reynolds Number. The Chezy resistance coefficient **C** is plotted as a function of Reynolds number in Chart 631. An auxiliary scale of Darcy resistance coefficient f is also shown for alternative use by the designer. The method of plotting is a form of the Moody diagram (Sheet $224-1$). The resistance equations for smooth and rough flow based on Keulegan's results and recommended by Chow¹ are given and plotted in Chart 631. The rough flow limit based on Rouse's pipe flow criterion) is also shown. The Keulegan constants were used in the Colebrook-White equation (Chart 224-1) for the transition flow zone. The Reynolds number used for plotting is

$$
R_e = \frac{4VR}{V}
$$

where $v =$ the kinematic viscosity.

The use of this form of the Reynolds number is recommended in the ASCE task force report.⁴

6. Basic Data. The plotted data in Chart **631** are for concrecelined channels. Both tranquil- and rapid-flow data are presented. The tranquil-flow data were computed from U. S. Army Engineer Waterways Experiment Station (WES) laboratory tests in brushgd-concrete flumes⁶,7 and from field tests results compiled by Scobey. 8.9 More recently obtained U. S. Bureau of Reclamation (USBR)¹⁰ and Italian¹¹ field data have also been included. These data were selected on the oasis of accuracy of flow measurements and conditions of concrete channel lining. Tests at the University of Iowa¹² indicate that the energy loss in flows having Froude numbers greater than 1.6 becomes a function of the Froude number and density and size of roughness elements. Additional energy loss is caused by instability of the flow. The plotted data points based on prototype tests at the Fort Randall¹³ and Fort Peck¹⁴ spillway chutes are for rapid flow with Froude numbers exceeding the stability criterion. These data represent the only known available measurements at R_e numbers approaching 10^8 .

631 to 631-2 Revised 1-68

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7. Suggested Design Criteria.

- a. Resistance coefficients. The data plotted in Chart **631** can be used for guidance in the design of concrete-lined channels with subcritical velocities. Resistance coefficients for these channels generally are in the transition zone shown in the chart. The flow regime is seldom hydraulically smooth or fully rough and the resistance coefficient is usually a function of both the Reynolds number and the relative roughness. Chart **631-1** is a plot relating Chezy C, Manning's n, the equivalent roughness k_S , and the hydraulic radius. Theoretically it is only applicable to rough flow conditions. This chart should be useful for relating C and n for the design of channels with riprapped banks (Charts 631-4 and 631-4/1). The equation for n on Chart **631-1** was developed by solving the rough flow equation given in Chart **631** in terms of Manning's n.
- b. Equivalent roughness k_s . In the use of Chart 631, a value of ks (equivalent sand grain diameter) has to be specified for the prediction of resistance. The hydraulic roughness **ks** in pipe flow is dependent only on the type of construction or the surface finish specified. However, in open channel flow it includes the effects of secondary flow **re**sulting from boundary geometry and to a lesser extent the free water surface. Experimental data for correlation of surface texture, channel geometry, and the resulting hydraulic equivalent roughness k_S are very limited. However, considerable variation in the selected k_S value results in only small changes in the flow energy loss.
	- (1) The following tabulation presents average **ks** values resulting from different types of concrete forming and surface finishing. It is based on computations made from the open channel resistance data plotted in Chart 631.

(Continued)

(2) The tabulation above can be used for selecting design **ks** values if the concrete forming and surface finishing can be obtained with good assurance. For general design computations the following k_S values for concrete are suggested:

- * To prevent undesirable undulating waves, flow-depthto-critical depth ratios between 0.9 and 1.1 should be avoided.
- **(3)** The determination of the equivalent surface roughness for riprap channels, rubble masonry, or other large roughness protrusions should be based on some estimate of the mean protruslon, riprap, or rock size. Use of the **D50** (mean) size as k_s , based on equivalent sphere weight, is a good approximation for stone riprap.

8. Application. Chart 631-2 is a sample computation sheet illustrating the use of Charts 631 and 631-1.

9. References.

- **(1)** Chow, V. T., Open-Channel Hydraulics. McGraw-Hill Book Co., Inc., New York, N. Y., 1959, pp 109-123.
- (2) U. S. Geological Survey, Roughness Characteristics of Natural Channels, by H. H. Barnes, Jr. Water-Supply Paper 1849, Washington, D. C., 1967.
- (3) Keulegan, G. H., "Laws of turbulent flow in open channels." Journal of Research, National Bureau of Standards, vol 21, Research Paper No. **1151** (December 1938), pp 707-741.

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- (4) Progress Report of the Task Force on Friction Factors in Open Channels, "Friction factors in open channels." **ASCE,** Hydraulics Division, Journal, vol 89, HY 2, paper 3464 (March 1963), pp 97-143.
- (5) Rouse, H., Engineering Hydraulics; Proceedings of the Fourth Conference, Iowa Institute of Hydraulic Research, June 12-15, 1949.
John Wiley & Sons, Inc., New York, N. Y., 1950, p 404.
- (6) U. S. Army Engineer Waterways Experiment Station, CE, Roughness Standards for Hydraulic Models; Study of Finite Boundary Roughness in Rectangular Flumes, by Irene E. Miller and Margaret S. Peterson. Technical Memorandum No. 2-364, Report **1,** Vicksburg, Miss., June 1953.
- (7) **_,** Hydraulic Capacity of Meandering Channels in Straight Floodways; Hydraulic Molel Investigation, by E. B. Lipscomb. Technical Memorandum No. 2-429, Vicksburg, Miss., March 1956.
- [(8) **U.** S. Department of Agriculture, The Flow of Water in Flumec. by 1 F. C. Scobey. Technical Bulletin No. 393, Washington, D. **C.,** December 1933.
- (9) **_,** The Flow of Water in Irrigation and Similar Canals, by F. C. Scobey. Technical Bulletin No. 652, Washington, D. C., February 1939.
- (10) U. S. Bureau of Reclamation, Analyses and Descriptions of Capacity Tests in Large Concrete-Lined Canals, by P. **J.** Tilp and M. W. Scrivner. Technical Memorandum 661, Denver, Colo., April 1964.
- (11) Grassino, R., "Determination of roughness coefficients for Oimena Canal." L'Energia Elettrica, vol XL, No. 6 (June 1963), pp 429-436. Translation by Jan **C.** Van Tienhoven for U. S. Army Engineer Waterways Experiment Station, CE, Translation No. 65-3, Vicksburg, Miss., May **1965.**
- (12) Rouse, H., Koloseus, H. J., and Davidian, J., "The role of the Froude number in open-channel resistance." Hydraulic Research, Journal of the International Association for Hydraulic Research, vol **1,** No. 1 (1963), pp 14-19.
- (13) U. S. Army Engineer Waterways Experiment Station, CE, Flow in Chute Spillway at Fort Randall Dam; Hydraulic Prototype Tests, by C. J. Huval. Technical Report No. 2-716, Vicksburg, Miss., April 1966.
- (14) U. S. Army Engineer District, Omaha, Nebraska. (Unpublished memo-
randum on Fort Peck Spillway tests, 1951.)

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Concrete-lined channel **Equivalent roughness k**₅ **Shape,** trapezoidal Chezy **C** Invert slope **(S) =** 0.0004 **Base** width **B** Flow depth (D) = 12 ft Froude No. < 0.85 --. Side slope **= I** on 2 Check Manning's n 2 Water temperature **" 60** F Discharge **(Q) = 15,000 cfs** Construction, **rail-mounted** traveling forms

GIVEN: REQUIRED:

From tabulation of equivalent roughness (par. **7b(1), Sheets 631** to **631.2), k, 0.003 ft** From Chart **001.1,** v **=** 1.22 x **10- ⁵**ft² /sec at **60** F

TRIAL COMPUTATIONS

P **1.** Assume **base** width **B = 50 ft**

$$
V = \frac{Q}{Area} = \frac{15,000}{74 \times 12} = 16.9 \text{ ft/sec}
$$

SAREA 2
 SAREA 2 12
 SAREA Perimeter = $\frac{74 \times 12}{103 \times 7} = 8.57$ **ft**

$$
R_e = \frac{4VR}{\nu} = \frac{4(16.9)(8.57)}{1.22 \times 10^{-5}} = 4.75 \times 10^7
$$

$$
\frac{R}{k_s} = \frac{8.57}{0.003} = 2860
$$
 C = 148 (Chart 631)

 $V = C\sqrt{RS} = 148\sqrt{8.57 \times 0.0004} = 8.67$ *ft/sec <* 16.9 *ft/sec*

2. Assume base width B **= 110 ft**

$$
V = \frac{15,000}{134 \times 12} = 9.33 \text{ ft/sec} \qquad R = \frac{134 \times 12}{163.6} = 9.83 \text{ ft}
$$

\n
$$
R_e = \frac{4(9.33)(9.83)}{1.22 \times 10^{-5}} = 3.0 \times 10^7
$$

\n
$$
\frac{R}{k_x} = \frac{9.83}{0.003} = 3280
$$

\n
$$
C = 149 \text{ (Chart 631)}
$$

\n
$$
V = 149 \sqrt{9.83 \times 0.0004} = 9.34 \approx 9.33 \text{ ft/sec}
$$

3. Check Froude No. (F) and Manning's n

PREPARED BY U. S. AGNY ENTINEER VATERVAYS EXPERIMENT STATION, VICKSBURG, MISSISSIPPI

$$
F = \frac{V}{\sqrt{gD}} \text{ (wide channel)} = \frac{9.33}{\sqrt{g(12)}} = 0.48 < 0.85
$$
\n
$$
n = 0.0145 \text{ (Chart 631-1)}
$$

OPEN CHANNEL FLOW **RESISTANCE COEFFICIENTS** SAMPLE COMPUTATION HYDRAULIC **DESIGN** CHART **631-2** REV 1-66 WES 0-56

HYDRAULIC DESIGN CRITERIA sHEETs 63-4 **AND** 631-4/1 open CHANNEL FLOW

OPEN CHANNEL FLOW COMPOSITE ROUGHNESS EFFECTIVE **MANNING'S** n

> **1.** Tables of recommended roughness coefficients for use in the Manning formula for the solution of open channel flow problems have been published in references 1 and 2. Chow² includes recommended values for channels having different bed and bank materials. In wide, shallow channels the bed roughness effects predominate. Conversely, in narrow deep channels the bank roughness is the primary factor contributing to the flow energy losses.

2. Basic Data. Procedures for computing the effective roughness coefficient n to be used in the Manning formula for channels with different bed and bank roughnesses have been developed by Horton,³ Colebatch, Einstein,⁵ and the U. S. Army Engineer District, Los Angeles, California.⁶ In each case the effective n value is a function of the bed and bank deep channels the bank roughness is the primary factor contributing to
the flow energy losses.
2. <u>Basic Data</u>. Procedures for computing the effective roughness
coefficient n to be used in the Manning formula for channels area. In their simplest form, the equations for effective n values can be written as

> (Los Angeles District) (1) $n_{\text{eff}} = \frac{\Sigma n A}{\Sigma A}$

$$
n_{\text{eff}} = \left[\frac{\Sigma(n^{3/2} \text{ P})}{P}\right]^{2/3} \quad \text{(Horton or Einstein)} \quad (2)
$$

$$
n_{\text{eff}} = \left[\frac{\Sigma(n^{3/2} A)}{\Sigma A}\right]^{2/3} \qquad \text{(Collection)} \quad (3)
$$

A and P are the channel flow subareas and wetted perimeter segments, respectively; n is the respective Manning roughness coefficient for each segment considered.

3. Study of the equations given in paragraph 2 indicates that for channels with smooth inverts and rough banks, use of the Horton-Einstein equation results in more conservative design t an use of either the Colebatch or the Los Angeles District equation. Laboratory and field investigations are needed for complete evaluation of the equations. The use of the Horton-Einstein equation is suggested for design purposes pending availability of additional test data.

4. For rectangular or trapezoidal channels, equation 2 can be written in the form \sim

$$
n_{eff} = \left(\frac{n_1^{3/2} P_1 + 2n_2^{3/2} P_2}{P_1 + 2P_2}\right)^{2/3}
$$
 (4)

where the subscripts **1** and 2 refer to the bed and bank wetted perimeters, respectively. The terms are further defined in the sketch in Hydraulic Design Chart 631-4/1.

5. Application. Chart 631-4 provides a rapid graphical method for determining the solution of equation 2 to obtain an effective n value for use in the design of uniform channel sections with different \cdot bed and bank roughnesses. The ordinates of the chart indicate the bed, bank, and combined effective roughness coefficients. The abscissas are values of the ratio of the bed and bank wetted perimeters. The effective n value is determined in the following manner. The chart is entered vertically from the bottom with the given value of $~2P_2/P_1~$ to its intersection with an imaginary line connecting n_1 and n_2 . The value of n_{eff} at this point is read on the right side of the chart.

6. Chart 631-4/1 can be used to obtain the required wetted perimeter ratio for use with Chart $631-4$. Chart $631-4/1$ presents bank-bed wetted perimeter relations for trapezoidal and rectangular channel sections as functions of the bed width, flow depth, and bank slope. These charts can be used with Charts 631 and 631-1 for the design of channels with riprapped banks.

7. References.

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- (2) Chow, V. T., Open-Channel Hydraulics. McGraw-Hill Book Co., Inc., New York, N. Y., 1959, Tables 5 and 6, p 111.
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- (4) Colebatch, G. T., "Model tests on Liawenee Canal roughness coefficients." Transactions of the Institution, Journal of the Institution of Engineers, vol 13, No. 2, Australia (February 1941), pp 27-32.
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(6) U. *S. Army,* Office, Chief of Engineers, Hydraulic Design **of** Flood Control Chaamels. EM **1110-2-1601** (unpublished Engineer Manual

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HYDRAULIC DESIGN CRITERIA

SHEET 660-1

CHANNEL CURVES

SUPERELEVATION

1. Purpose. Flows in curved channels result in increases in depth along the outside channel walls with corresponding decreases along the inside walls. The difference in the water-surface elevations between the channel center line and the outside wall is called the flow superelevation. This rise in water surface is a function of the channel shape, velocity, width, and radius of curvature. Chart 660-1 presents a graphical means of estimating superelevation for various combinations of channel velocities, widths, and radii of curvature.

2. Design Controls. Channel capacity (wall heights) should be based on the maximum expected resistance (friction) factor. The curve geometry and flow superelevation should be based on the minimum expected resistance factor. This design combination should result in economically conservative design for all flows.

3. Design Equations. The transverse rise in water surface of flow in a channel bend can be adequately described for both tranquil and rapid flow using an equation adapted from the centrifugal force equations.

$$
\Delta y = C \frac{V^2 W}{gr} \tag{1}
$$

where

- Δy = the rise (superelevation plus surface disturbances) in water surface between the channel center line and the outside wall, ft
	- $C = a$ coefficient depending upon flow Froude number, channel shape, and curve geometry
	- $V = average channel velocity, fps$
- $W =$ straight channel water-surface width, ft
- $g =$ acceleration of gravity, ft/sec²
- $r =$ radius of curvature at center line, ft

The following tabulation relates the coefficient C with flow conditions, channel shape, and curve geometry. These relations are also shown by the sketches in Chart 660-1.

+. Curve Design.

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- a. Tranquil flow. The required increase in the outer wall height in a channel curve over that of the straight channel for both rectangular and trapezoidal channels is obtained from Chart 660-1 using a C value of 0.5. The inner wall height should remain that of the straight channel. The unbalanced flow condition in the curve causes helicoidal flow that can result in undesirable scour and deposition in and downstream from the curve. Tests by Shukryl indicate that helicoidal flow can be minimized if the curve radius is greater than three times the channel width.
- b. Rapid flow. Rapid flow in a simple circular curve results in a transverse rise in the water surface approximately twice that occurring with tranquil flow. This increase results from surface disturbances generated by changes in dixection. These disturbances persist for many channel widths downstream **i** of the curve. Superelevation for rapid flow can be estimated from Chart 660-1 using the appropriate C values given in the tabulation above or in the chart. A detailed analysis of the cross waves generated in simple curves is given by Ippen.²

The criterion for minimum radius of a simple curve, based on structures built by the Los Angeles District, is:

$$
r_{\min} = \frac{4v^2w}{gy} \tag{2}
$$

with y equal to the flow depth for the minimum expected friction factor (Chart 631). This criterion is recommended for rapid flow curves with or without invert banking. A similar criterion for maximum allowable superelevation for acceptable flow conditions in rectangular channels is

$$
\Delta y_{\text{max}} = 0.09 \text{W} \tag{3}
$$

660-1

- c. Invert banking. Invert banking maintains flow stability in curved channels and when used with spiral transitions results in minimum total rise in water surface between the channel center line and outside wall. It is limited to channels of rectangular cross sections. The invert is usually banked by rotating the bottom about the channel center line. The invert along the inside wall is depressed by **6y** below the center-line elevation with a corresponding rise along the outside wall. The banking upstream and downstream from the curve should be accomplished linearly in accordance with the spiral transition lengths determined from equation **3** of Sheets 660-2 to 660-2/4. Wall heights on both sides of banked curves are usually designed to be the same as the wall height of the straight channel. Banking of trapezoidal channels is not practicable. Such channels should be designed wherever possible to have long radius curves resulting in minimum superelevation.
- 5. References.

p 400-6-588. the 400-6-58

- (1) Shukry, A., "Flow around bends in an open flume." Transactions, American Society of Civil Engineers, vol **115,** paper 2411 **(1950))** pp 751-779.
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HYDRAULIC DESIGN CRITERIA **SHEETS 660-2** TO 660-2/4 **CHANNEL** CURVES WITH SPIRAL TRANSITIONS

RAPID FLOW

1. Purpose. Spiral transitions are used to provide gradual change in channel curvature for rapid flow entering and leaving circular bends.¹ The compound circular curve has also been used for this purpose.² Use of spiral transitions eliminates the surface disturbances discussed in Sheet 660-1 and minimizes required wall height increases or channel banking.

2. Spiral Transitions. Spiral curves involve the solution of cubic equations by complex procedures, extensive successive approximation, or computers. The Los Angeles District (LAD) has prepared extensive spiral tables for easier manual design of rapid flow channels.³ HDC 660-2 to $660 - 2/4$ summarize these tables and illustrate their application to channel design.

3. The LAD spiral is a modification of Talbot's railroad spiral and consists of a series of compounded circular arcs of 12.5-ft lengths. The spiral has varying radii, decreasing in finite steps from the beginning of the spiral. The curve geometry, equations, and the definitions used to develop the LAD tables are given in Chart 660-2. Two equal spirals are shown, one upstream and one downstream of the circular curve. The central angle of the first arc (δ_1) establishes the shape of the spiral. The central angle subtended by a spiral of n number of arcs is given by:

$$
\Delta s = n^2 \delta_1 \tag{1}
$$

where

 t_h Δs = total central angle at the n^{21} arc of the spiral, sec

n **=** number of arc lengths of 12.5 ft each

 δ_1 = central angle of the first arc, sec

4. Unbanked Curves. The minimum length of spiral recommended by Douma4 for an unbanked curve is

$$
L = 1.82 \frac{W}{\sqrt{gy}}
$$
 (2)

660 · 560-2/4

where V and y are the velocity and flow depth, respectively, computed using a minimum resistance coefficient (Chart 631) and W is the water-
surface width.

5. Banked Curves. The minimum spiral length recommended by Gildea and Wong⁵ for banked curves is:

$$
L = 30\Delta y \tag{3}
$$

where **Ay** is the rise in water surface between the channel center line and the outside wall. Use of this criterion will not usually result in free drainage of a channel banked by rotating the invert about the center-line elevation.

6. Unequal Spirals. Unequal spiral lengths at the beginning and end of the circular curve may be required to meet special field conditions. The geometric relations between the spirals and the circular curve are given in Chart $660-2/1$. With these relations determined, the design for each spiral proceeds as in the case of equal spirals.

7. Spiral Design Tables. The original LAD tables have been abridged and are presented in Chart $660 - 2/2$. The chart should be adequate for design purposes and for preparation of contract drawings. Values of spiral lengths L , tangent distances X , and offsets Y are tabulated for n number of stations for 22 spirals. The method of computing values of X and Y , and the radius r of the central simple curve is given in reference 3. The curve number corresponds to the value of the first spiral arc angle δ_1 , in sec, and indicates the rate of change in curvature. The minimum spiral length should be that which satisfies equation 2 (unbanked) or 3 (banked), provides optimum fit to local physical conditions, and is commensurate with economy of construction.

8. Application. The computation procedure for a banked invert curve with spiral transitions at each end is given in Chart $660-2/3$. The final curve layout for the example is given in Chart 660-2/4. In cases of intermittent flow the banking may result in an undesirable pool of stagnant water along the inside wall. This can be avoided by selecting a longer downstream spiral. The length of this spiral is dependent upon the curve number selected and the number of spiral arc lengths required to attain a radius approximating that computed for the central curve. Twice the spiral length multiplied by the channel slope must equal or exceed the invert banking for free drainage.

9. Computer Program. A computer program for the design and field layout of the channel curve geometry is given in Appendix V of EM lll0-2-160l.°

660-2 to 660-2/4

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10. References.

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- (2) Ippen, A. T., and Knapp, R. T., Experimental Investigations of Flow in Curved Channels. Reproduced by U. S. Army Engineer Office, Los Angeles, Calif. (2 volumes), 1958 (abstract of Results and Recommendations).
- (3) U. S. Army Engineer District, Los Angeles, CE, Modified Spiral Curve Tables, June 1948.
- (4) Douma, J. H., Discussion of "High-velocity flow in oren channels; A symposium." Transactions, American Society of Civil Engineers, vol 116, paper 2434 (1951), pp 388-393.
- (5) Gildea, A. P., and Wong, R. F., "Flood control channel hydraulics." Proceedings, Twelfth Congress of the International Association for Hydraulic Research, 11-14 September 1967, vol **1** (1967), pp 330-337.
- (6) U. S. Army, Office, Chief of Engineers, "Appendix V: Computer program for designing banked curves for supercritical flow in rectangular channels," Engineering and Design; Hydraulic Design of Flood Control Channels. EM 1110-2-1601, Washington, D. C., 1 July 1970.

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S Design **Q 15,000** cfs Channel width W **= 50** ft Invert slope $S = 0.005$ Curve deflection angle **I** 45 **deg** Channel **shape -**rectangular Design controls **-** Sheets **631** to **631-2,** par **7b(2)**

REQUIRED:

Spiral (minimum length) and simple curve (minimum radius) geometries with invert banking

COMPUTE:

a. Simple curve radius **(min)**

$$
r_{\min} = \frac{4V^2W}{gy} = \frac{4(29.05)^2(50)}{(32.2)(10.33)} = 507.42 \text{ ft} \quad (\text{Eq 2, Sheet 660-1})
$$

b. Approximate banking (Chart $660-1$) = $2 \Delta y$

$$
\frac{r}{W} = \frac{507.42}{50} = 10.14
$$

For V = 29.05 fps and
$$
\frac{r}{W}
$$
 = 10.14 ; $\frac{\Delta y}{C}$ = 2.6
 Δy = 2.6(0.5) = 1.3 ft

- c. Spiral length **(min)** L $L = 30 \Delta y = 30(1.3) = 39$ ft **(Eq 3)**
- **d.** Spiral curve geometry

For $r_{min} \approx 507$ and $L \approx 39$ u.s spiral curve No. 520 (Chart 660-2/2, Sheet 5 of 5) $\Delta_{\rm s} = 02^{\circ}18^{\rm t}40^{\rm m}$

***** $\delta_n = (2n - 1)\delta_1$ (Chart 660-2)

CHANNEL CURVE EXAMPLE **COMPUTATION**

HYDRAULIC **DESIGN** CHART **660-2/3**

(SHEET I OF 2)

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e. Simple curve geometry (use r **620 ft)**

(1) Central angle 0 (Chart **660.2)** $\theta = 1 - 2\Delta_s = 45 - 2(02^{\circ}18'40'')$ $= 45 - (04°37'20'') = 40°22'40''$

(2) Curve length **Lc** (Chart **660-2)**

$$
L_c = \frac{(1 - 2\Delta_s)r}{57.2958} = \frac{(40°22'40'')(620)}{57.2958}
$$

$$
= \frac{40.38(620)}{57.2958} = 436.95 \text{ ft}
$$

- **f.** Total curve length **L^c** $L_T = 2L + L_c = 2(50) + 436.95 = 536.95$ ft
- **g.** Corrected invert banking **= 2Ay**

$$
\frac{r}{W} = \frac{620}{50} = 12.40
$$

For V = 29.05 fps and $\frac{r}{W} = 12.40$
 $\frac{\Delta y}{C} = 2.2$ (Chart 660-1)
 $\Delta y = 2.2C = 2.2(0.5) = 1.10$ ft
 $2\Delta y = 2.20$ ft

h. Maximum allowable Δy_{max}

 $2\Delta y_{max} = 0.18W = 0.18(50) = 9.0$ ft $(Eq 3,$ Sheet 660-1) $\Delta y_{\text{max}} = 4.5$ ft > $\Delta y = 1.10$ ft (item g) OK

i. Curve tangent distance T_s

 $T_s = X - r \sin \Delta_s + (Y + r \cos \Delta_s) \tan \frac{1}{2}$ 49.992 **- 620** sin(02018140") **+ (0.693 + 620** cos 02018'40") **tan 22030'00"** 49.992 **- 620(0,04033) + [0.693 + 620(0.99919)]** 0.41421

49.992 **- 25.005 + (0.693 +** 619.498)0.41421

24.987 **+** (620.191)0.41421 **= 281.87** ft

CHANNEL CURVE EXAMPLE COMPUTATION

HYDRAULIC **DESIGN** CHART **660-2/3 (SHEET** 2 OF 2)

EXPARED BY U.S. ANNY ENGINEER VATERVAYS EXPERIMENT STATION VICKSBURG, MISSISSIPPI

HYDRAULIC DESIGN CRITERIA

SHEET 703-1

RIPRAP PROTECTION

TRAPEZOIDAL **CHANNEL, 60 DEG-BEND**

BOUNDARY SHEAR DISTRIBUTION

1. Riprap used to aid in the stabilization of natural streams and art'.ficial channels is most commonly placed in the vicinity of bends. Procedures for estimating the required size of riprap in straight channels have been presented by the U. S. Army Engineer Waterways Experiment Station¹ and Office, Chief of Engineers.² No similar procedure has been developed for evaluating riprap size for channel bends. Hydraulic Design Chart 703-1 is based on laboratory tests at the Massachusetts Institute of Technology $(MT)^3$ and should be useful for estimating relative boundary shear distribution in simple channel bends having trapezoidal cross sections, moderate side slopes, and approximately 60-deg deflection angles. It may also serve as a general guide for riprap gradation in natural channel bends of similar geometry. Shear distribution diagrams for other bend geometries and flow conditions have been published.^{3,4}

2. Laboratory studies of boundary shear in open channel bends of trapezoidal cross section^{3,5} indicate that the highest boundary shear caused by the bend geometry occurs immediately downstream from the bend and along the outside bank. Another area of high boundary shear is located at the inside of the bend. The relative boundary shear distribution in a simple bend with a rough boundary is given in Chart 703-1. The chart is based on fig. 21 of the MIT report.³

3. Experimental Data. Laboratory tests on smooth channel bends have been made at MIT, 3 at U. S. Bureau of Reclamation, 5 and at the University of Iowa.⁶ In addition, limited tests on rough channel bends have been made at MIT. In the latter tests, the channel was roughened by fixing 0.18- by **0.10-** by 0.10-in. parallelepipeds to the boundary in a random manner which resulted in an absolute roughness height of **0.10** in. The MIT test channel was 24 in. wide with **1** on 2 side slopes. The boundary shear distribution patte'n has been generally found to be the same in all tests on simple curves having smooth and rough boundary conditions. However, the magnitude of the ratio of bend local boundary shear to the average boundary shear in the approach channel appears to be a'function of the channel and bend geometry. Some work has also been done at MIT3 on boundary shear distribution in double and reverse curve channels.

4. Application. Extensive variation in riprap gradation throughout a bend may not be practical or economical. However, increasing the 50 percent rock size and the thickness of the riprap blanket in areas of expected high boundary shear is recommended. Chart 703-1 can be used as a

guide for defining the location and extent of these areas in simple channel bends. The boundary shear ratios should be less than those shown in Chart 703-1 for bends with smaller deflection angles or with larger ratios of bend radius to water-surface width (r/w) .

5. References.

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- (2) U. S. Army Engineer, Office, Chief of Engineers, Stone Riprap Protection for Channels, by S. B. Powell.
- (3) Ippen, A. T., and others, Stream Dynamics and Boundary Shear Distributions for Curved Trapezoidal Channels. Report No. 47, Hydrodynamics Laboratory, Department of Civil Engineering, Massachusetts Institute of Technology, Cambridge, January 1962.
- (4) Ippen, A. T., and Drinker, P. A., "Boundary shear stress in trapezoidal channels." **ASCE,** Hydraulics Division, Journal, vol 88, HY *5,* paper **3273** (September 1962), pp 143-179.
- (5) U. S. Bureau of Reclamation, Progress Report No. 1--Boundary Shear Distribution Around a Curve in a Laboratory Canal, by **E.** R. Zeigler. Hydraulics Branch Report No. HYD 526, 26 June 1964.
- (6) Yen, Ben-Chie, Characteristics of Subcritical Flow in a Meandering Channel. Institute of Hydraulic Research, University of Iowa, Iowa City, 1965.

703-1

V HYDRAULIC DESIGN CRITERIA t **SHEET** ⁷⁰⁴

ICE THRUST ON HYDRAULIC STRUCTURES

1. The expansion of an ice sheet as the result of a rise in air temperature can develop large thrusts against adjacent structures. The magni tude of this thrust is dependent upon the thickness of the ice sheet, the rate of air temperature rise, the amount of lateral restraint, and the extent of direct penetration of solar energy. Ice pressures from 3350 to 30,000 lb per lin ft⁽¹⁾ have been used for design purposes. EM 1110-2-**2200 ?3)** suggests a unit pressure of not more than 5000 **lb** per sq ft of contact area and indicates that ice thickness in the United States will not normally exceed 2 ft.

2. Although the work of Rose⁽²⁾ stimulated a number of studies on ice pressure, the graphs proposed by him are of value for design purposes. These graphs are reproduced in HDC 704.

2. Although the work of Rose⁽²⁾ stimulated a number of studies on
ice pressure, the graphs proposed by him are of value for design purposes.
These graphs are reproduced in HDC 704.
3. The ice thrust curves in HDC 704 are are presented to show the effects of lateral restraint and solar radiation. The expected ice thicknesses, air temperature rise, and possible snow blanket thickness are dependent upon geographical location and elevation above sea level. In the region of chinook winds rapid air temperature rises can occur. The U. S. Weather Bureau has recorded a 49° F rise in two minutes at Spearfish, S. Dak. When the ice sheet is confined by steep banks close to the structure, spillway piers, or other vertical restrictions, the criteria for complete lateral restraint should be used. The direct effects of solar energy on the thrust are eliminated when the ice sheet is insulated by a blanket of snow only a few inches thick.

4. References.

- **(1)** American Society of Civil Engineers, "Ice pressure against dams: A symposium." Transactions, American Society of Civil Engineers, vol **119** (1954), **pp 1-42.**
- (2) Rose, **E.,** "Thrust exerted by expanding ice sheet." Transactions, American Society of Civil Engineers, vol 112 (1947), pp **871-900.**
- **(3) U.** S. *Army,* Office, Chief of Engineers, Engineering and Design, Gravity Dam Design. EM 1110-2-2200, 25 September 1958.

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HYDRAULIC DESIGN CRITERIA

SHEET 711

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DISCHARGE COEFFICIENTS

1. Purpose. Several monoliths of the spillway section of a concrete gravity dam are occasionally left at a low elevation during spillway construction for diversion of floodflows. Information on the discharge characteristics of these monoliths is necessary for determining the number of monoliths required to allow floodflows to pass safely. **HDC** 711 should serve as a guide for selection of discharge coefficients for this purpose.

2. Free Overflow. The flow over low concrete monoliths is generally treated as flow over a broad-crested weir. The equation for free discharge is:

$$
Q = C_e (L - 2 KH) H^{3/2}
$$

where C_f is an empirical coefficient, L is the length of opening transverse to the flow, H is the head on the weir, and K is an end contrac- **_I,** tion coefficient. The value of K is conventionally taken to be **0,10** for square-end contractions. The free-flow coefficient C_f varies with the ratio of head H to width B of the broad-crested weir in the direction of flow. **HDC** 711a shows the variation of **Cf** with H/B resulting from investigations summarized by Tracy.^1 Kindsvater² has recently shown the effect of boundary layer development on broad-crested-weir discharge. The rate of development is a function of the bottom roughness. However, present knowledge of this effect does not justify considering boundary layer development for diversion flow computations. The curve resulting from the classical experiments of Bazin³ as shown by the solid curve in HDC 711a is recommended for general design purposes.

3. Submergence Effect. Discharge coefficients for broad-crested weirs are not usually affected until the depth of submergence is about 0.67 or more of the head on the weir. The phenomenon is commonly expressed in terms of the ratio of the coefficient of the submerged weir to that of the unsubmerged weir C_S/C_P as a function of the ratio of the tailwater depth on the weir to the head on the weir H_2/H_1 **.** Available data indicate that sharp-crested-weir coefficients are more sensitive to submergence than broad-crested-weir coefficients.

4. Available data on the effects of submergence on discharge coefficients for both sharp- and broad-crested weirs^{2,4,5,6} are summarized in IHDC 711b. As far as is known, rectangular broad-crested weirs have not been subjected to submergence tests. A suggested design curve for submerged low monoliths is given in the chart.

5. Applicqtion. The suggested design curves given in HDC 711 should, serve as guides for estimating diversion flows over low monoliths. In cases where the head-discharge relation may be critical, a more exact rela- j tion should be obtained by hydraulic model investigation. A model study of proposed low-monolith diversion schemes for Allatoona Dam^{ℓ} was made because of critical diversion requirements.

6. References.

- (1) Tracy, H. J., Discharge Characteristics of Broad-Crested Weirs. U. S. Geological Survey Circular 397, 1957.
- (2) Kindsvater, C. E., Discharge Characteristics of Embankment-Shaped Weir; Studies of Flow of Water Over Weirs and Dams. U. S. Geological Survey Water-Supply Paper 1617-A, 1964.
- (3) Bazin, M. H., "Experiences nouvelles sur l'ecoulement en diversoir." Annales des Ponts et Chaussees, vol 7, Series 7, 1896.
- (4) U. S. Geological Survey, Weir Experiments, Coefficients, and Formulas, by R. E. Horton. Water-Supply Paper No. 200, 1907, p 146.
- (5) King, H. W., Handbook of Hydraulics for the Solution of Hydraulic Problems, revised by E. F. Brater, 4th ed. McGraw-Hill Book Co., Inc., New York, N. Y., 1954, pp 4-18.
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- (7) U. S. Army Engineer Waterways Experiment Station, CE, Sluices and Diversion Scheme for Allatoona Dam, Etowah River, Georgia; Model Investigation. Technical Memorandum No. 214-2, Vicksburg, Miss., November 1946.

Service Construction

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HYDRAULIC DESIGN CRITERIA

SHEET 712-1

STONE STABILITY

VELOCITY **VS STONE** DIAMETER

1. Purpose. Hydraulic Design Chart **712-1** can be used as a guide for the selection of rock sizes for riprap for channel bottom and side slopes downstream from stilling basins and for rock sizes for river closures. Recommended stone gradation for stilling basin riprap is given in paragraph 6.

2. Background. In 1885 Wilfred Airy¹ showed that the capacity of a stream to move material along its bed by sliding is a function of the sixth power of the velocity of the water. $^{\text{1}}$ Henry Law applied this concept to the overturning of a cube, ² and in 1896 Hooker² illustrated its application to spheres. In 1932 and 1936 Isbash published cpefficients for the stability of rounded stones dropped in flowing water. $3,4$ The design curves given in Chart 712-1 have been computed using Airy's law and the experimental coefficients for rounded stones published by Isbash.

3. Theory. According to Isbash the basic equation for the movement of stone in flowing water can be written as:

$$
V = C \left[2g \left(\frac{\gamma_s - \gamma_w}{\gamma_w} \right) \right]^{1/2} (D)^{1/2}
$$
 (1)

where

 $V =$ velocity, fps **C =** a coefficient $g =$ acceleration of gravity, ft/sec² γ_{s} = specific weight of stone, lb/ft^{3} Y_{yz} = specific weight of water, $1b/ft^3$ D **=** stone diameter, ft

The diameter of a spherical stone in terms of its weight W is

$$
D = \left(\frac{6W}{\pi \gamma_s}\right)^{1/3} \tag{2}
$$

^fSubstituting for D in equation **1** results in

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$$
v = c \left[2g \left(\frac{v_s - v_w}{v_w} \right) \right]^{1/2} \left(\frac{6w}{\pi v_s} \right)^{1/6}
$$

which describes Airy's law stated in paragraph 2.

4. Experimental Results. Experimental data on stone movement in flowing water from the early (1786) work of DuBuat⁷ to the more recent Bonneville Hydraulic Laboratory tests⁰ have been shown to confirm Airy's law and Isbash's stability coefficients.^{7} The published experimental data are generally defined in terms of bottom velocities. However, some are in terms of average flow velocities and some are not specified. The Isbash coefficients are from tests with essentially no boundary layer development and the average flow velocities are representative of the velocity against stone. When the stone movement resulted by sliding, a coefficient of 0.86 was obtained. When movement was effected **by** rolling or overturning, a coefficient of 1.20 resulted. Extensive U. S. Army Engineer Waterways Experiment Station laboratory testing for the design of riprap below stilling basins indicates that the coefficient of 0.86 should be used with the average flow velocity over the end sill for sizing stilling basin riprap because of the excessively high turbulence level in the flow. For impacttype stilling basins, the Bureau of Reclamation⁸ has adopted a riprap design curve based on field and laboratory experience and on a study by Mavis and Laushey.⁹ The Bureau curve specifies rock weighing 165 lb/ft³ and is very close to the Isbash curve for similar rock using a stability coefficient of **0.86.**

5. Application. The curves given in Chart 712-I are applicable to specific stone weights of 135 to 205 lb/ft³. The use of the average flow velocity is desirable for conservative design. The solid-line curves are recommended for stilling basin riprap design and other high-level turbulence conditions. The dashed line curves are recommended for river closures and similar low-level turbulence conditions. Riprap bank and bed protection in natural and artificial flood-control channels should be designed in accordance with reference **10.**

- 6. Stilling Basin Riprap.
	- a. Size. The W50 stone weight and the **D50** stone diameter for establishing riprap size for stilling basins can be ob-
tained using Chart 712-1 in the manner indicated by the heavy arrows thereon. The effect of specific weight of the rock on the required size is indicated by the vertical spread of the solid line curves.
	- b. Gradation. The following size criteria should serve as guidelines for stilling basin riprap gradation.
		- (1) The lower limit of W50 stone should not be less than the weight of stone determined using the appropriate "Stilling Basins" curve in Chart 712-1.

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- (2) The upper limit of W₅₀ stone should not exceed the **AV** weight that can be obtained economically from the quarry or the size that will satisfy layer thickness requirements as specified in paragraph 6c.
- (3) The lower limit of W_{100} stone should not be less than two times the lower limit **of** W50 stone.
- (4) The upper limit of W100 stone should not be more than five times the lower limit of W_{50} stone, nor exceed the size that can be obtained economically from the quarry, nor exceed the size that will satisfy layer thickness requirements as specified in paragraph 6c.
- **(5)** The lower limit of Wl5 stone should not be less than onesixteenth the upper limit of W100 stone.
- (6) The upper limit of W15 stone should be less than the upper limit of W50 stone as required to satisfy criteria for graded stone filters specified in EM 1110-2-1901.
- (7) The bulk volume of stone lighter than the W15 stone should not exceed the volume of voids in the revetment without this lighter stone.
- (8) Wo to W₂₅ stone may be used instead of W₁₅ stone in criteria **(5), (6),** and (7) if desirable to better utilize available stone sizes.
- c. Thickness. The thickness of the riprap protection should be 2D50 max or **l.5DloO** max , whichever results in the greater thickness.
- d. Extent. Riprap protection should extend downstream to where nonerosive channel velocities are established and should be placed sufficiently high on the adjacent bank to provide protection from wave wash during maximum discharge. The required riprap thickness is determined by substituting values for these relations in equation 2.

7. References.

- (1) Shelford, W., "On rivers flowing into tideless seas, illustrated by
the river Tiber." <u>Proceedings, Institute of Civil Engineers</u>, vol 82
(1885).
	- (2) Hooker, **E.** H., "The suspension of solids in flowing water." Transactions, American Society of Civil Engineers, vol 36 (1896), pp 239- 34o,
	- (3) Isbash, S. V., Construction of Dams by Dumping Stones in Flowing

~ **712-1** Revised 9-70 Water, Leningrad, 1932. Translated by A. Dorijikov, U. S. Army Engineer District, Eastport, CE, Maine, 1935.

- (4) ________, "Construction of dams by depositing rock in running water." Transactions, Second Congress on Large Dams, vol 5 (1936), **pp 123-136.**
- (5) DuBuat, P. L. G., Traite d'Hydraulique. Paris, France, 1786.
- (6) U. S. Army Engineer District, Portland, CE, McNary Dam Secc',d Step Cofferdam Closure. Bonneville Hydraulic Laboratory Report No. 51-1, **1956.**
- (7) U. S. Army Engineer Waterways Experiment Station, CE, Velocity Forces Y on Submerged Rocks. Miscellaneous Paper No. 2-265, Vicksburg, Miss., April 1958.
- (8) U. S. Bureau of Reclamation, Stilling Basin Performance; An Aid in Determining Riprap Sizes, by A. J. Peterka. Hydraulic Laboratory Report No. HYD-409, Denver, Colo., 1956.
- (9) Mavis, F. T. and Laushey, L. M., **"A** reappraisal of the beginning of bed movement **-** competent velocity." Second Meeting, International Association for Hydraulic Structure Research, Stockholm, Sweden, 1948. See also Civil Engineering, vol 19 (January 1949), pp 38, 39, and 72.
- **(10)** U. S. Army, Office, Chief of Engineers, Engineering and Design; Hydraulic Design of Flood Control Channels. EM 1110-2-1601, Washington, D. C., 1 July 1970.

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HYDRAULIC DESIGN CRITERIA SHEETS 722-1 TO 722-3 STORM DRAIN OUTLETS **FIXED** ENERGY DISSIPATORS

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1. Purpose. Storm drains frequently terminate in unstable channels and gullies. Under these conditions dissipation of the energy of the outflow is required to prevent serious erosion and potential undermining and subsequent failure of the storm drains. Adequatg energy dissipation can be accomplished by extensive riprap protection $^{\ast}_{\cdot}$ or by construction of specially designed fixed energy dissipators. $3,4,5,6$

2. Hydraulic Design Charts (HDC's) 722-1 to -3 present design criteria for three types of laboratory tested energy dissipators.³ Each type has its advantages and limitations. Selection of the optimum type and size is dependent upon local tailwater conditions, maximum expected discharge, and economic considerations.

3. Stilling Wells. The stilling well energy dissipator shown in HDC 722-1 was developed at the U. S. Army Engineer Waterways Experiment Station (WES).³ Energy dissipation in this stilling well is relatively independent of tailwater and is accomplished by flow expansion in the well, by impact of the fluid on the base and wall of the well, and by the change in momentum resulting from redirection of the flow to vertically upward. WES laboratory tests³ indicated that the structure performs satisfactorily for flow-pipe diameter ratios $(Q/D_0^2, 5)$ up to 10 with a well-pipe diameter ratio of 5.

4. HDC 722-1 shows the relation between storm drain diameter, well diameter, and discharge. Designing for operation beyond the limits shown in HDC 722-1 is not recommended. Intermediate ratios of stilling welldrain pipe diameters within the limits shown in HDC 722-1 can be computed using the equation given in this chart.

Impact Energy Dissipators. The U. S. Bureau of Reclamation (USBR)⁵ has developed an impact energy dissipator which is an effective stilling device even with deficient tailwater. The dimensions of this energy dissipator in terms of its width are shown in HDC 722-2. Energy dissipation in the basin is accomplished by the impact of the entering jet on the vertically hanging baffle ana by the eddies that are formed following impact on the baffle.

6. HDC 722-2 shows the relation between storm drain diameters, basin width, and discharge. WES laboratory tests³ showed that this structure properly designed performs satisfactorily for Q/D_0^2 .⁵ ratios up to 21. Intermediate ratios of basin widths within the limits shown in HDC 722-2 can be computed using the equation given in this chart. Design for operation beyond these limits is not recommended. The WES

tests also showed that optimum energy dissipation for the design flow 4 occurs with the tailwater midway up the hanging baffle. Excessive tailwater should be avoided as this causes flow over the top of the baffle.

7. Hydraulic Jump Energy Dissipators. The St. Anthony Falls Hydraulic Laboratory (SAFHL)⁶ has developed the hydraulic jump energy dissipator shown in HDC 722-3. Design equations for dimensionalizing the structure in terms of the square of the Froude number of the flow entering the dissipator are also given in the chart. WES laboratory tests³ showed that this type of stilling basin performs satisfactorily for ratios of Q/D_0^2 ⁵ up to 9.5 with a basin width three times the storm drain diameter. WES tests were limited to basin widths of **1,** 2, and 3 times the drain diameter with drops (drain invert to stilling basin) of **0.5** and 2 times the drain diameter. Parallel stilling basin walls were used for basin width-drain diameter ratios of 1 and 2. The transition wall flare was continued through the basin for $W = 3D_0$. Parallel basin sidewalls are generally recommended for best performance. Transition sidewall flare $(1:D')$ during the WES tests was fixed at 1 on 8. The invert transition to the stilling basin should conform to the geometry of the trajectory of a flow not less than 1.25 times the drain outlet portal design velocity.

8. HDC 722-3 shows the relation between storm drain diameter and discharge for stilling basin widths up to 3 times the drain diameter which results in satisfactory performance. WES tests have been restricted to the limits shown in HDC 722-3, and the equation g_{reen} in the chart can be used to compute intermediate basin width-drain diameter ratios within those limits. General WES model tests of outlet works indicate that this equation also applies to ratios greater than the maximum shown in the chart. However, outlet portal velocities exceeding 60 fps are not recommended for designs containing chute blocks. This chart does not reflect the outlet invert transition effects on basin performance. The design of the basin itself (HDC 722-3) is dependent upon the depth and velocity of the flow as it enters the basin. The values should be computed taking into account the drain outlet transition geometry.

9. Riprap Protection. Riprap protection in the immediate vicinity of the energy dissipator is recommended. Preliminary, unpublished WES test results3 on riprap protection below energy dissipators indicates the following average diameter (D_{50}) stone size should result in adequate erosion protection.

$$
D_{50} = D \left(\frac{V}{\sqrt{gD}}\right)^3
$$

where

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D50 = the minimum average size of stone, ft, whereby 50 percent by weight of the graded mixture is larger than **D50** size

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- D = depth of flow in outlet channel, ft
- V **=** average velocity in outlet channel, ft
- $g =$ gravitational acceleration, $ft/sec²$

10. References.

- (1) U. S. Army Engineer Waterways Experiment Station, CE, Erosion and Riprap Requirements at Culvert and Storm-Drain Outlets; Hydraulic Laboratory Model Investigation, by J. P. Bohan. Research Report H-70-2, Vicksburg, Miss., January 1970.
- (2) , Practical Guidance for Estimating and Controlling Erosion at Culvert Outlets, by B. P. Fletcher and J. L. Grace, Jr. Miscellaneous Paper H-72-5, Vicksburg, Miss., May 1972.
- (3) ._ , Evaluation of Three Energy Dissipators for Storm-Drain Outlets; Hydraulic Laboratory Investigation, by J. L. Grace, Jr., and G. A. Pickering. Research Report H-71-1, Vicksburg, Miss., April 1971.
- (4) , Impact-Type Energy Dissipator for Storm-Drainage Outfalls Stilling Well Design; Hydraulic Model Investigation, by J. L. Grace, Jr. Technical Report No. 2-620, Vicksburg, Miss., March 1963.
- (5) Beichley, G. L., Progress Report No. XIII Research Study on Stilling Basins, Energy Dissipators and Associated Appurtenances - Section 14, Modification of Section **6** (Stilling Basin for Pipe or Open Channel Outlets - Basin VI. Report No HYD-572, Hydraulics Branch, Division of Research, U. S. Bureau of Reclamation, Denver, Colo., June **1960.**
- (6) Blaisdell, F. W., The SAF Stilling Basin. Agricultural Handbook No. 156, Agricultural Research Service and St. Anthony Falls Laboratory, University of Minnesota, Minneapolis, Minn., April 1959.

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HYDRAULIC DESIGN CRITERIA SHEET 722-4 TO 722-7 STORM DRAIN OUTLETS

RIPRAP ENERGY DISSIPATORS

1. Purpose. Criteria for the hydraulic design of fixed energy dissipating structures for storm drain outlets are presented in Hydraulic Design Charts (HDC's) 722-1 to 722-3. Under some conditions adequate energy dissipation can be accomplished more economically using riprap as an alternate to fixed structures. HDC's 722-4 to 722-5 present three basic riprap energy dissipator designs developed at $WES.1.2$

2. Scour Holes. Scour holes at storm drain exit portals effectively dissipate flow energy and reduce downstream erosion. However, uncontrolled scour holes can undermine the storm drain with subsequent structural failure. Basic laboratory tests were conducted at $WES¹$ during the period 1963-1969 to investigate scour hole development and erosion protection in cohesionless material downstream from storm drain exit portals. These tests showed that the length, width, depth, and volume of the scour hole could be related in terms of the storm drain diameter D_o in feet, the discharge Q in cfs, and the flow duration t in minutes. The tailwater depth TW in feet over the storm drain invert was also found to be important. The following set of design equations² describes the basic scour hole dimensions for two controlling tailwater conditions.

$$
\frac{L_{sm}}{D_o} = C \left[\left(\frac{Q}{D_o^2 \cdot 5} \right)^{O \cdot 71} \left(t^{O \cdot 125} \right) \right] \tag{1}
$$

$$
\frac{D_{\rm sm}}{D_{\rm o}} = C \left[\left(\frac{Q}{D_{\rm o}^2 \cdot 5} \right)^{0.375} \left(t^{0.10} \right) \right]
$$
 (2)

$$
\frac{W_{\rm sm}}{D_{\rm o}} = c \left[\left(\frac{Q}{D_{\rm o}^2 \cdot 5} \right)^{0.915} \left(t^{0.15} \right) \right]
$$
 (3)

$$
\frac{V_s}{D_o^3} = C \left[\left(\frac{Q}{D_o^2 \cdot 5} \right)^2 \left(t^0 \cdot 375 \right) \right]
$$
 (4)

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where

Lsm = scour hole length, **ft** D_{sm} = depth of maximum scour, ft W_{sm} = half the width of the hole at the location of maximum scour, ft V_s = volume of material removed from scour hole, ft³

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Empirically determined values of C in the equations above for the two controlling tailwater conditions are:

3. HDC 722-4 shows dimensionless scour hole profiles and cross sections for the two limiting tailwater conditions.

4. Horizontal Riprap Blanket. HDC 722-5 shows the recommended length Lsp and geometry of the horizontal riprap blanket protection required for satisfactory dissipation of the energy of the design outflow from a storm drain. (The required D_{50} riprap size can be estimated using HDC 722-7.)

5. Preformed Scour Holes. Laboratory studies have shown that satisfactory energy dissipation of storm drain outflow occurs in ripraplined, preformed scour holes of nominal size. **HDC** 722-6 shows the recommended design for preformed scour holes 0.5 and **1.ODo** deep. The D50 minimum stone size required for each scour hole depth can be estimated using HDC 722-7.

5. Application. Study of the basic test data indicates that the resulting design criteria are generally applicable to both circular and rectangular conduits flowing full or partly full. For rectangular conduits the conduit width is used in place of the diameter **Do** of the circular conduits.

6. References.

- (1) U. S. Army Engineer Waterways Experiment Station, CE, Erosion and Riprap Requirements at Culvert and Storm-Drain Outlets; Hydraulic Model Investigation, by J. P. Bohan. Research Report H-70-2, Vicksburg, Miss., January 1970.
- (2) ________, Practical Guidance for Estimating and Controlling Erosion at Culvert Outlets, by B. P. Fletcher and J. L. Grace, Jr., Miscellaneous Paper H-72-5, Vicksburg, Miss., May 1972.

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HYDRAULIC DESIGN CRITERIA

SHEET 733-1 SURGE TANKS

THIN PLATE ORIFICES

HEAD LOSSES

1. Thin plate orifices are often used in surge tank risers to restrict the flow during load-on and load-off operations. Computation of the head losses through these orifices is of interest in the design of surge tanks.

2. A number of experiments have been made on head losses through orifices in straight pipe. When an orifice is placed in a surge tank riser close to the penstock tee, the energy loss of flow entering or leaving the riser is affected by the orifice flow. Indri's⁽²⁾ extensive study of orifices in branches has made available new data on head loss coefficients considered to be applicable to surge tank problems. The pipe used in this study was 9 cm (3.54 in.) in diameter. The orifice plates were located in the branches 125 mm (4.92 in.) from the center line of the main pipe. The test results indicate that the combined tee and orifice loss coefficients were independent of Reynolds number for $R_e > 3 \times 10^4$.

3. HDC **733-1** presents a head loss coefficient curve for thin plate orifices in tees. The head loss coefficient is based on the combined tee and orifice head loss. Indri's data shown in this chart indicate that a single curve is applicable to load on-load off turbine conditiong. Also shown in this chart are head loss coefficient curves by Weisbach(3) and Marchetti⁽¹) for thin plate orifices in straight pipe. These curves indicate that the location of the orifice with respect to other disturbances affects the head loss.

4. The data in HDC **733-1** are based on the equation:

$$
H_{L} = K_{o} \frac{v^{2}}{2g}
$$

where

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 H_T = head loss across the orifice or orifice and tee, ft K_0^- = head loss coefficient \bar{V} = velocity in riser, ft per sec

The head loss coefficient is plotted as a function of the ratio of the square of the riser diameter D to the square of the orifice diameter d . A sketch of an orifice in a straight pipe is included in the chart for purposes of defining the terms involved.

5. References.

- **(1)** Caric, D. M., "Tehnicka hydraulika." Gradevenska, Knjiga, Belgrad (**(1952).**
- (2) Indri, E., "Richerche sperimentali su modelli di strozzature per pozzi piezometrici (Experimental research on models of constrictions for surge tanks)." L'Energia Elettrica, vol 34, No. 6 (June 1957), pp 554-569. Translation by Jan C. Van Tienhoven, for U.S. Army Engineer Waterways Experiment Station, CE, Translation No. 60-3, Vicksburg, Miss., April 1960.

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(3) Weisbach, J., Untersuchungen in den Gebieten der Mechanik und Hydraulik. Leipzig, 1945.

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