

**AFAPL-TR-79-2103** 



# HIGH BYPASS TURBOFAN COMPONENT DEVELOPMENT

Phase II-Fan Detail Design

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SEFCRE COMPLETING FORM 19) REPORT DOCUMENTATION PAGE 2. GOVY ACCUSSION NO. 1. PECISIENT'S CATALOG NUMBER I. REPORT AFAPLHTR-79-2103 omponer TYPE OF REPORT A PERIOD COVERED Final Keport DEVELOPMENT, <u>HIGH BYPASS TURBOFAN</u> Dec 1978-0cr 1979 DETAIL DESIGN . 7. AUTHOR(s) D. C. Chapman F33615-78-C-2Ø14 PROGRAM ELEMENT, PROJECT, TASK 9. PERFORMING ORGANIZATION NAME AND ADDRESS 16 Detroit Diesel Allison, Division of General Motors Corporation, P.O. Box 894, Indianapolis, Indiana 3066-15-24 46206 11. CONTROLLING OFFICE NAME AND ADDRESS Air Force Aero Propulsion Laboratory, Air Force December 1979 Wright Aeronautical Laboratories, Air Force Systems 68 Command, Wright-Patterson Air Force Base, Ohio 45483 15. SECURITY CLASS. (of this report) Unclassified 154. DECLASSIFICATION/ DOWNGRADING SCHEDULE 16. DISTRIBUTION STATEMENT (of this Report) Approved for Public Release; Distribution Unlimited 17. DISTRIBUTION STATEMENT (of the abetract entered in Block 20, if different from Report) 18. SUPPLEMENTARY NOTES AFAPL-TR-79-2034, High Bypass Turbofan Component Development, Phase I--Preliminary Design and Life Reference report -Phase I Cycle Cost Analysis of Candidate Engines, D. C. Chapman and W. A. Redmond 19. KEY WORDS (Continue on reverse side if necessary and identity by block number) Turbofan Engine, Trainer Enginer, Fan, Small Fan 20. ABSTRACT (Continue on reverse side if necessary and identify by block number) The objective of this program is to develop an advanced, small, high bypass turbofan component applicable to an advanced, fuel-efficient engine for a new Air Force primary trainer aircraft. The program consists of two phases. Phase I consisted of the preliminary design and life cycle cost analysis of candidate engines. This report documents Phase II which was the detail design of the selected fan configuration. Primary design point of the fan is 1.3 pressure ratio at 55.9 lbm/sec flow. At a lower speed, the fan will produce 1.65 (cont) DD , FORM 1473 EDITION OF 1 NOV 65 IS GREAL ETT

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#### Preface

This report was prepared by D. C. Chapman of Detroit Diesel Allison, Division of General Motors Corporation, Indianapolis, Indiana.

Design details of the fan stage designed in Phase II of Contract No. F33615-78-C-2014, High Bypass Turbofan Component Development, sponsored by the A.F. Aero Propulsion Laboratory are presented. The Air Force contract monitor was Capt. Larry Gill.

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#### SUMMARY

An advanced technology fan stage adaptable to a small, high bypass turbofan engine for a future Air Force primary trainer has been designed. Primary design point is at 1.8 pressure ratio and 55.9 lb/sec flow. At a lower speed, this fan stage will produce a 1.65 pressure ratio at a flow compatible with the GMA500 core engine to form a high bypass engine meeting all requirements for the trainer application.

At the 1.8 pressure ratio design condition, the fan rotor operates at 1606 ft/sec tip speed with an inlet annulus specific flow of 42.3 lbm/sec/ft<sup>2</sup>. The inlet hub/tip radius ratio is 0.40. The rotor has 20 blades of multiple circular arc airfoil sections with an aspect ratio of 1.64. Maximum thickness-to-chord ratio varies nonlinearly from 8.5% at hub to 2.5% at tip.

Forty-two vanes of multiple circular arc cross section are tilted rearward at the tip to increase blade to vane spacing for noise considerations. The vane aspect ratio is 2.32 and the maximum thickness-to-chord ratio varies from 6% at the hub to 8% at the tip.

The design meets all structural design requirements including steady and vibratory stresses, blade flutter, and bird ingestion. Substantial margins exist for blade and wheel permanent set at 122% speed and for burst at 130% speed. Adequate margins also exist for low cycle fatigue, considering 12,000 cycles to design speed with  $K_{\rm c}=3$  at blade leading and trailing edges,  $K_{\rm c}=1.4$  at blade crown,  $K_{\rm c}=1.4$  in the wheel rim, and  $K_{\rm c}=2.0$  in the wheel web. Allowable blade vibratory stress exceeds the required  $\pm 15$  ksi at resonance points and the required  $\pm 5$  ksi at nonresonance points.

The blading was also checked for torsional stall flutter and found to be satisfactory.

Bird ingestion requirements of Mil-E-5007D forced a slight thickening of the blade leading edge region in conjunction with a material change from titanium to stainless steel.

The predicted noise levels are substantially below FAR Part 36 levels at takeoff and approach. Furthermore, the ground idle noise levels are greatly reduced from those of the current trainer configuration.

## SECTION I INTRODUCTION

The current Air Force primary trainer (T37) fleet is approaching the end of its useful life, and a replacement aircraft will be needed. A fuel-efficient engine for the replacement aircraft must be developed. Advanced technology engines, such as the GMA500, suitable for the core of such a high bypass engine are being developed. Advanced technology fan stages in this size class are not available. To fill this void in technology, Detroit Diesel Allison (DDA), Division of General Motors Corporation, has conducted the High Bypass Turbofan Component Development Program for the United States Air Force Aero Propulsion Labratory, Wright-Patterson AFB, Ohio. The program consisted of two phases:

- Phase I—Preliminary Design and Life Cycle Cost Analysis of Candidate Engines
- Phase II--Detailed Design of the Fan stage chosen from Phase I

The Phase I studies were reported in Report No. AFAPL-TR-79-2034, High Bypass Turbofan Component Development, Phase I--Preliminary Design and Life Cycle Cost Analysis of Candidate Engines by D. C. Chapman and W. A. Redmond. This report documents the design of the fan stage completed in Phase II of the program.

# Section II DESIGN REQUIREMENTS

The core engine of choice in the Phase I selection process was the GMA500, an advanced technology turboshaft engine which was a winner in the United States Army Advanced Technology Demonstrator Engine (ATDE) Competition. DDA is currently under contract to complete 500 hours of ruming on the GMA500 engine starting early in CY 1979. The engine consists of a two-stage centrifugal compressor, foldback annular combustor, two-stage gasifier turbine, and two-stage power turbine.

In Phase I of this program, fans of 1.5, 1.65, 1.8, and 2.0 pressure ratio were matched with the GMA500 core engine to form candidate high bypass ratio engines for system life cycle cost analysis. The performance of all engines met or exceeded the requirements of this contract. Using representative aircraft characteristics, these engines, designated PD418, were applied to the mission requirements established by the Air Force. Both aircraft gross weight and system life cycle cost were minimized with a fan pressure ratio of 1.65, although the advantage of that pressure ratio over 1.5 and 1.8 pressure ratios was not great. DDA, therefore, recommended to the Air Force that the 1.65 pressure ratio fan be selected for detail design in Phase II of the program.

The Air Force identified a higher technology level with the 1.8 fan pressure ratio and because the Life Cycle Cost (LCC) penalty was small, selected that pressure ratio for Phase II. DDA preferred the 1.65 pressure ratio not only because of the LCC analysis but because the engine sea level static thrust level was approximately 7.5% greater than the engine with 1.8 fan pressure ratio. A mutually agreeable set of design conditions were established wherein the fan would be designed to achieve 1.65 pressure ratio at a flow compatible with the GMA500 core engine and also to achieve 1.8 pressure ratio at a higher flow and speed. The primary design point thus established is:

- Pressure ratio 1.8:1
- Corrected flow 55.86 lbm/sec
- Efficiency 85%

The secondary design point, which matches the GMA500 core engine requirement at 25,000 ft, 0.5 Mach number is:

• Pressure ratio 1.65:1

• Corrected flow 52.8 lbm/sec

• Efficiency 87%

Structurally, the fan should meet the requirements of Mil-E-5007D, including bird ingestion capability. Furthermore, the fan noise levels should be within the limits of FAR Part 36 at both takeoff and approach. An unofficial goal was to achieve a substantial reduction in noise at ground idle compared to the existing primary trainer.

# SECTION III AERODYNAMIC DESIGN

The aerodynamic design is presented at the 1.8 fan pressure ratio operating condition. Flow-path and vector diagram details are followed by rotor and stator blading information.

#### FLOW-PATH AND VECTOR DIAGRAMS

The design parameters for the small high bypass fan are:

• Stage pressure ratio	1.8:1
• Corrected flow rate, lbm/sec	55.86
• Adiabatic efficiency, %	85.2
• Rotor inlet hub/tip radius ratio	0.40
• Corrected tip speed, ft/sec	1606
• Corrected speed, rpm	21685
• Mechanical speed, rpm	20223
• Corrected specific flow rate,	
lbm/sec/ft <sup>2</sup>	42.28

The velocity diagrams of the fan were obtained using the DDA Axial Compressor Design System. A description of the design system is given in Appendix A.

The fan was designed at an altitude cruise condition of 25,000 ft at 0.5 Mach number. This point represents the maximum mechanical speed achieved by the fan in a representative trainer mission.

A schematic of the fan flow-path is shown in Figure 1. The fan has a constant tip diameter of 16.974 in. and a rotor inlet hub-to-tip radius ratio of 0.40. The rotor hub ramp angle is 31.25 deg. The number of rotor airfoils is 20 while the stator has 42 vanes. The number of vanes and the vane-blade spacing were consequences of acoustical considerations.

The average value of the blade inlet absolute Mach number is 0.617. The blade inlet relative Mach numbers are supersonic for the outer 75% of the span. The exit relative Mach numbers are all subsonic (Figure 2). The average vane exit Mach number is 0.465. The inlet and exit Mach number profiles for the vane are shown in Figure 3.

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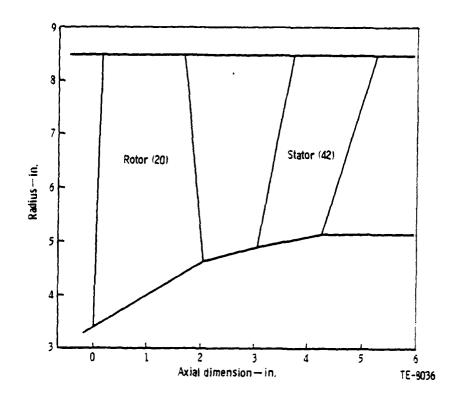


Figure 1. Schematic of fam flow path.

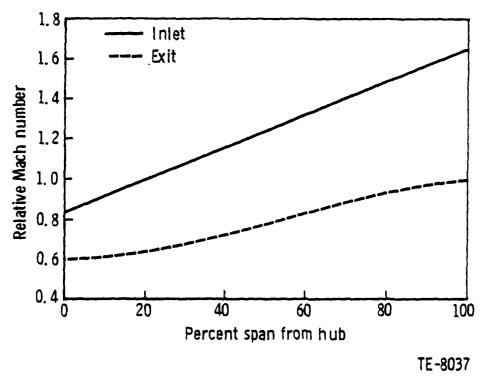


Figure 2. Blade Mach numbers.

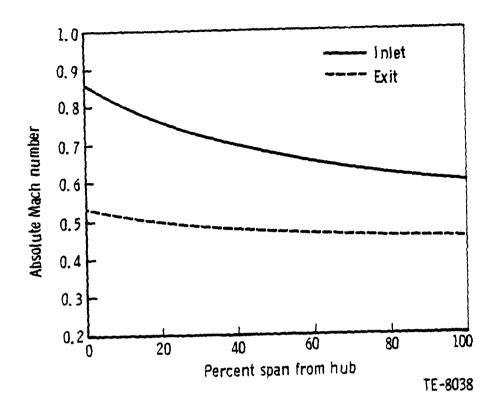


Figure 3. Vane Mach numbers.

The predicted blade and vane total pressure loss coefficients are illustrated in Figure 4. The resulting average efficiencies are 88.4% for the blade and 85.2% for the stage. The spanwise distribution of the design point loadings (diffusion factors) are shown in Figure 5. They are moderately high but the estimated surge margin for the fan is 18.6%. This surge margin estimate is based on a correlation of blade aspect ratio, relative Mach number, and tip loading at surge for various single stage compressors (Figure 6).

Figure 7 shows the blade inlet and exit relative air angles while Figure 8 is a plot of vane turning angles. The exit air angle from the vane is designed to be 0.0 degrees (axial).

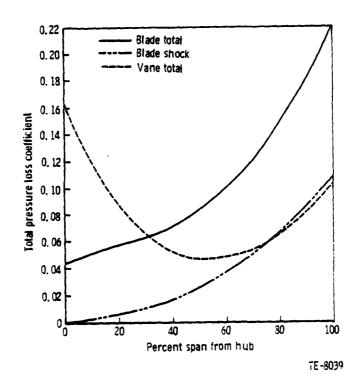


Figure 4. Radial distribution of total pressure loss coefficient.

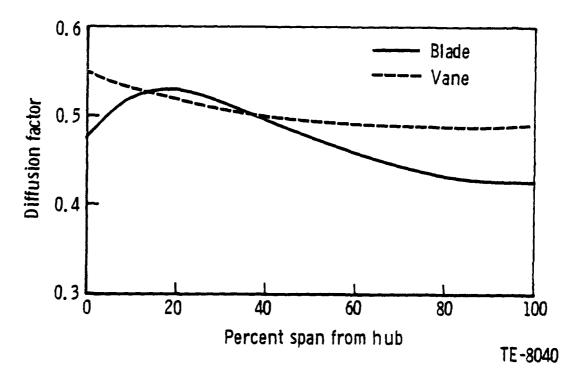


Figure 5. Blade and vane loading distributions.

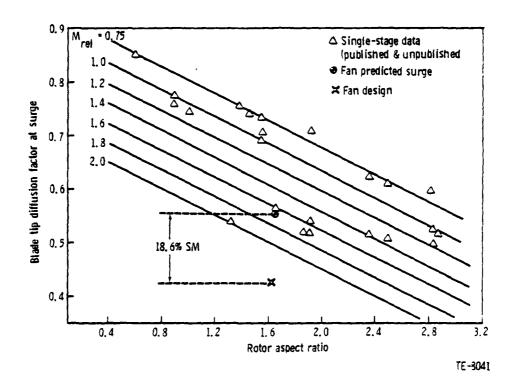


Figure 6. Single stage surge margin correlation.

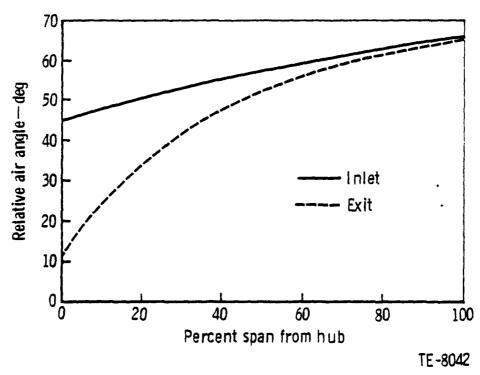


Figure 7. Blade air angles.

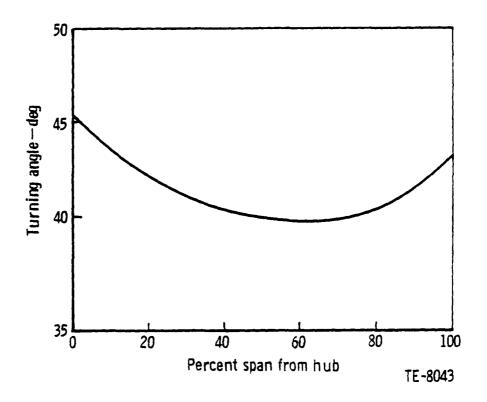


Figure 8. Vane turning angles.

The design point vector diagrams, calculated along streamlines, are tabulated for the blade and vane leading and trailing edge stations in Appendix B.

#### BLADE DESIGN

There are 20 blades with an aspect ratio of 1.64 (based on average span and true mean chord). The blade consists of multiple circular arc (MCA) airfoil sections designed on conical surfaces approximating streamlines of revolution. An MCA airfoil is shown schematically in Figure 9. It is made up of two circular arcs which define three metal angles: inlet  $(\beta_1^*)$ , exit  $(\beta_2^*)$ , and inflection  $(\beta_1^*)$ . A metal angle is the angle between the axial direction and the mean camber line at a specified location. A blade section is designed by adjusting the metal angles to satisfy incidence, deviation, and starting

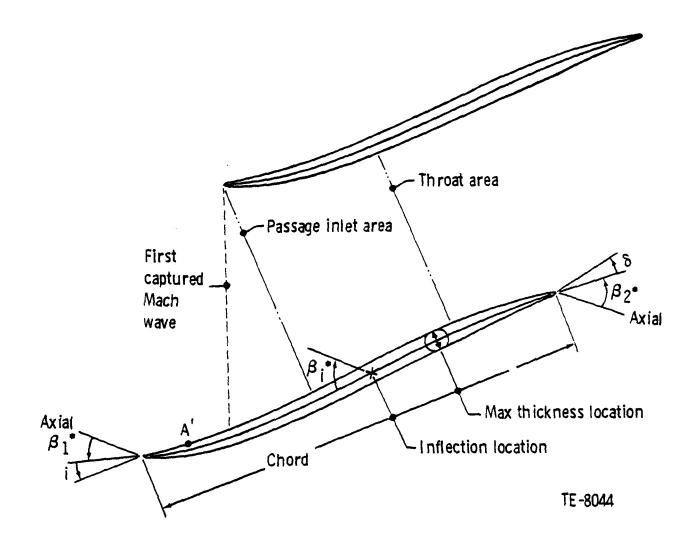


Figure 9. MCA airfoil definitions.

margin criteria. In the outer portion of the fan blade, where the inlet relative Mach number is supersonic, the airfoils were shaped to minimize shock loss. In the subsonic region of the blade, the airfoil shape transitions from the first supersonic section down to a near double-circular arc airfoil section at the hub.

A low aspect ratio, and, therefore, a long average chord, was selected to meet flutter criteria without the use of part-span shrouds. The spanwise chord taper was selected to satisfy the solidity requirements and also be viable from a weight and stress standpoint. The radial distributions of chord and solidity for the blade are shown in Figures 10 and 11, respectively. The maximum thickness to chord ratio (Figure 12) was set to avoid responsive resonant conditions and to maintain radial uniformity of blade mechanical properties. One of the mechanical considerations in the design was blade integrity with bird ingestion. The leading edge radius of the blade was set at 0.0125 in. from the hub to 60% span and then tapered to 0.010 in. at the tip. This maximized blade strength in the primary impact area while at the same time minimizing the efficiency penalty in the high inlet Mach number area at the blade tip from increased shock loss.

-

For the portion of the blade which has supersonic relative inlet Mach numbers, incidence was set on the suction surface at a point halfway between the leading edge and the emanation point of the first captured Mach wave (point A' of Figure 9). This incidence is the offset of the suction surface from a "free" streamline, which would exist if there were no blade forces, and it establishes the maximum flow the cascade can pass when the throat is not the limiting factor. The incidence value was set at 1.5 deg and is intended to account for leading edge blockage, suction surface boundary layer, and the bow shock wave. In the subsonic portion of the blade, the meanline incidence for each airfoil section was selected to locate the throat near the passage inlet.

Deviation angles were calculated using a modified form of the NACA 2-D rule for circular arc meanlines and then adding an empirical adjustment. The modification is a circulation correction based on the radius change of the streamline across the blade airfoil section.

The radial distributions of meanline incidence angle and deviation angle are shown in Figure 13. The resulting meanline blade angles are shown in Figure 14.

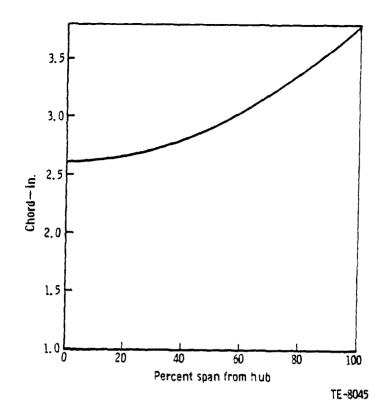


Figure 10. Blade chord.

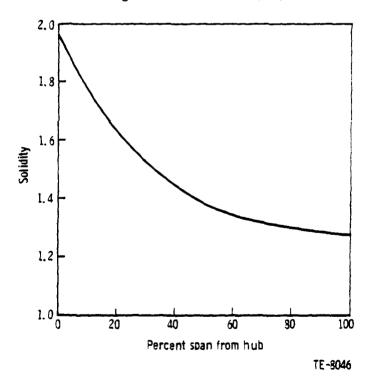


Figure 11. Blade solidity.

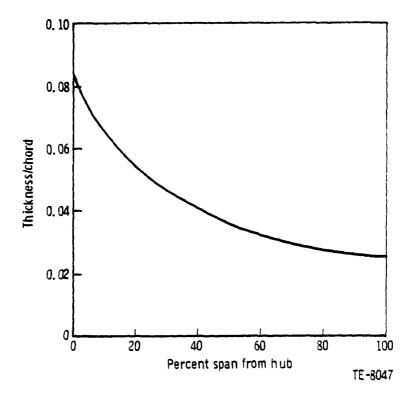


Figure 12. Blade maximum thickness/chord ratio.

The chordwise location of maximum thickness and circular arc inflection are shown in Figure 15. These were selected for two reasons: (1) to avoid accelerating suction surface curvatures ahead of the anticipated passage inlet shock wave location and (2) to set the passage inlet area and contour the airfoil passage between the passage inlet and the throat to minimize the velocity change through the passage. The design throat minimum critical area ratio (A/A\* min) distribution for the supersonic airfoil sections is set to 1.03 for a normal shock total pressure loss applied at the passage entrance with a linear distribution of profile loss from the leading to trailing edge of the airfoil section. Streamtube contraction and the effect of radius change are accounted for.

Figure 16 shows the blade hub, mean, and tip conical airfoil sections in engine orientation. For manufacturing purposes, the airfoil sections were redefined on planes normal to the stacking line. The stack line is a radial line passing through the center of gravity of the hub conical section. The blade manufacturing coordinates are listed in Appendix C with definitions given in Figure C-1.

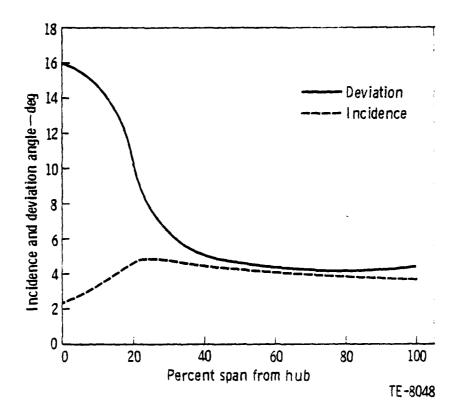


Figure 13. Blade incidence and deviation angles.

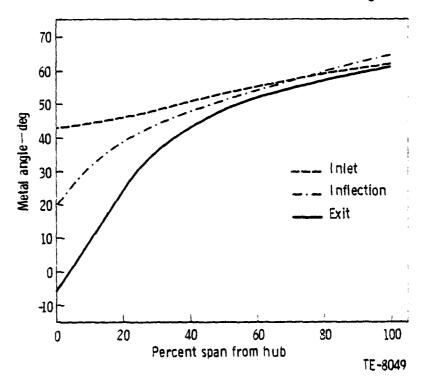


Figure 14. Blade metal angles.

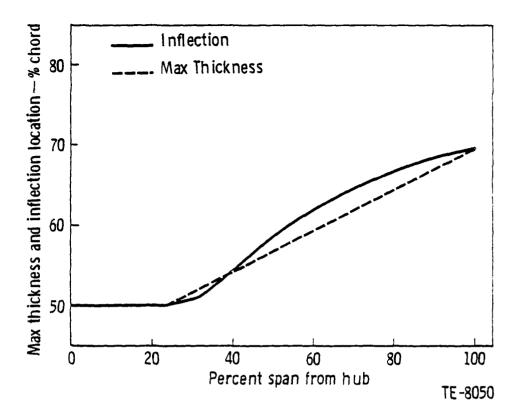


Figure 15. Blade maximum thickness and inflection locations.

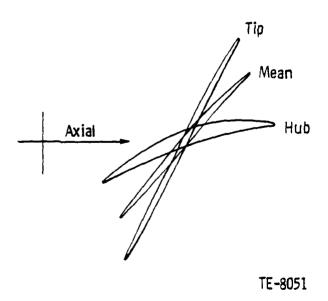


Figure 16. Blade conical airfoil sections.

#### VANE DESIGN

The vane is also made up of MCA airfoil sections. The vane axis is tilted rearward from radial at an angle of 13.5 deg. This gives a more desirable acoustic spacing between the blade and vane at the tip while minimizing hub length for bypass engine applications. There are 42 vanes with an aspect ratio of 2.32 and a solidity of 1.78 at the I.D. and 1.30 at the 0.D. (Figure 17). The radial distribution of chord is shown in Figure 18. The maximum thickness to chord ratio varies linearly from 6% at the hub to 8% at the tip.

The incidence angles were selected to position the throat location at the vane passage inlet. The passage throat margins were based on minimum loss cascade data (Figure 19). Deviation angles were determined from the NACA 2-D rule with an empirical adjustment. The incidence and deviation angles for the vane are presented in Figure 20. Vane metal angles are shown in Figure 21. Vane hub, mean, and tip conical airfoil sections are illustrated in Figure 22.

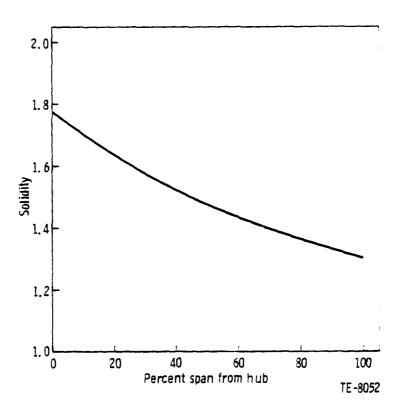


Figure 17. Vane solidity.

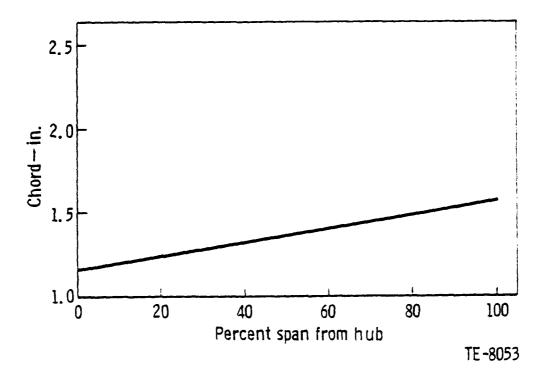


Figure 18. Vane chord.

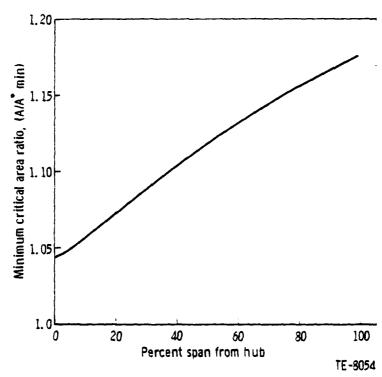


Figure 19. Vane passage throat minimum critical area ratio.

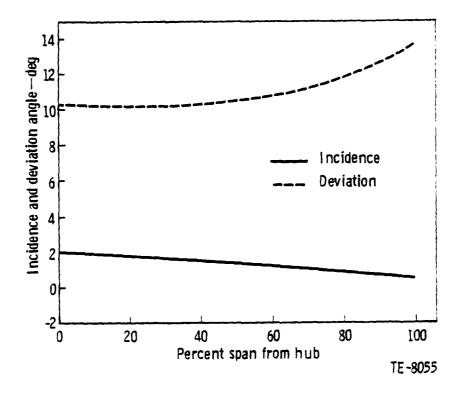


Figure 20. Vane incidence and deviation angles.

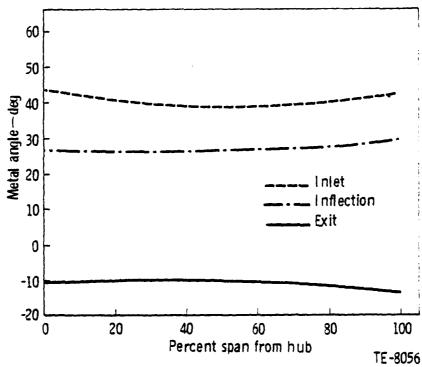


Figure 21. Vane metal angles.

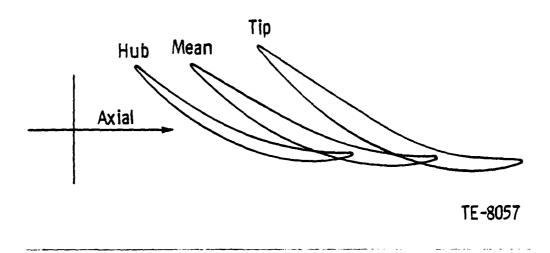


Figure 22. Vane conical airfoil sections.

The manufacturing coordinates for the vane are given in Appendix C with pertinent airfoil section definitions given on Figure C-2. The section coordinates were defined on planes normal to a stacking line. The stack line for the vane is on a radial line passing through the vane hub section c.g.

# SECTION IV STRUCTURAL ANALYSIS

Structural analysis of the fan rotor consisted of calculating airfoil and wheel steady-state stresses, airfoil vibrational characteristics, and bird ingestion capabilities. Satisfying all these requirements while retaining an aerodynamically acceptable configuration required numerous iterations on the design with changes in flow-path shape, spanwise chord and thickness distribution, and eventually a material change. The material change resulted from the inability to analytically satisfy bird ingestion requirements in an aerodynamically viable blade with titanium material. A stainless steel material (17-4 PH) was, therefore, substituted with consequent weight penalties. The final design meets or exceeds the requirements of Mil-E-5007D.

A titanium rotor (Ti-6Al-4V) design was near completion when the material change was deemed necessary. A satisfactory match-up of blade to wheel had not been obtained, and a valid hot-to-cold run was yet to be completed when the steel rotor analysis was started. Airfoil stresses reported here for the titanium rotor were determined with internal program boundary conditions for clamped hub, free tip. These automated constraints are not accurate for an integral blade-wheel rotor; preliminary blade/wheel match-up attempts indicated that airfoil crown stresses would be increased approximately 10 KSI over the reported results while leading edge and trailing edge stresses would be reduced.

The results of the analysis of the steel rotor are reported first. Titanium rotor results follow in a skeletonized format.

#### STEADY-STATE STRESSES

Design criteria for the rotor are given in Table 1. No steady-state blade stress will exceed 95% of the 0.2% yield strength of the material at 122% of design mechanical speed. High cycle fatigue requirements for the blade are a 15,000-psi allowable vibratory stress at resonance with a  $\rm K_{\rm L}=3.0$  at leading and trailing edges to allow for foreign object damage. Low cycle fatigue requirements for both wheel and blade are for greater than 12,000 start-stop cycles (zero-to-maximum stress) with a reliability of 0.9999. Finally, the wheel burst speed must exceed 130% of design speed.

# TABLE 1 Structural design criteria.

#### Blade

Permanent set

Low cycle fatigue

High cycle fatigue

95% 0.2% yield @ 122% speed
12,000 start-stop cycles
15 ksi allowable vibratory
stress at resonance. FOD (K
t
= 3.0) at leading and trailing
edge

### Wheel

Wheel burst

Low cycle fatigue

130% speed

12,000 start-stop cycles

Design point stress analyses were performed for the fan rotor at mechanical speeds of 20,223 and 18,950 rpm corresponding to 1.8 and 1.65 pressure ratios, respectively. The analysis is accomplished with finite element computer models which account for centrifugal loads, air loads, temperature effects, airfoil tilts, airfoil untwist, and wheel deflection. Airfoil bending stresses were minimized by tilting the airfoil in the direction of air loads.

#### Steel Rotor

Airfoil principal stress levels on both suction and pressure surfaces are shown for 1.8 and 1.65 pressure ratio design points in Figures 23 and 24, respectively. The maximum level of 110 ksi occurs near the airfoil hub on both suction and pressure surfaces at the higher pressure ratio and 100 ksi at the lower pressure ratio. The 110 ksi local principal stress on the blade surface compares with an average section stress of 61.5 ksi at design speed. To check the requirement for no damaging permanent set at 122% of design speed, it is

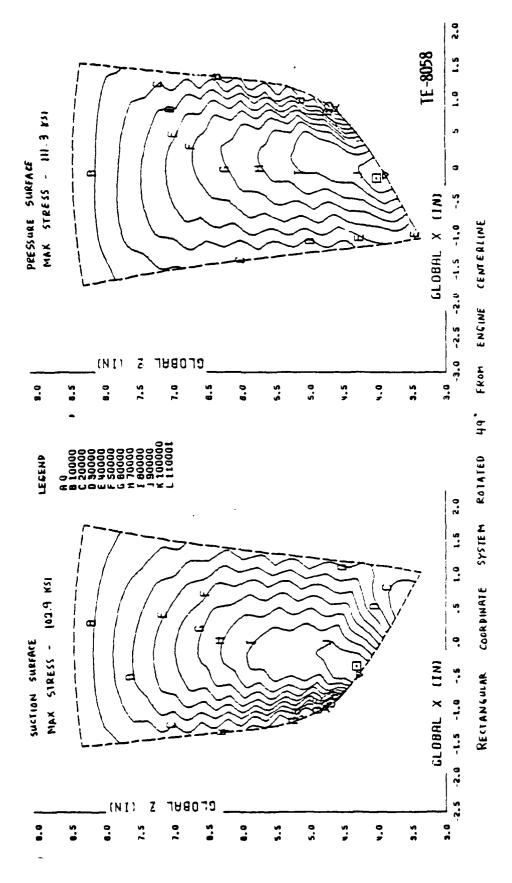


Figure 23. Principal blade stresses at 20,223 rpm.

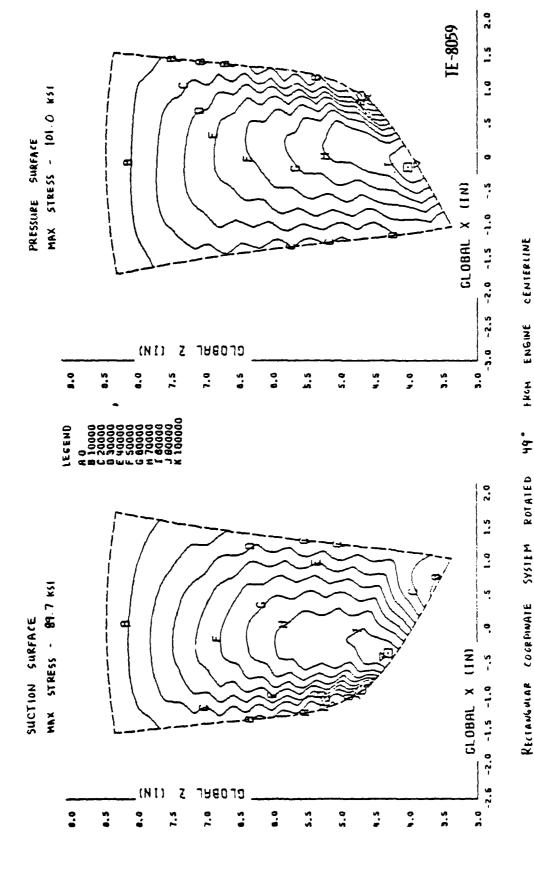


Figure 24. Principal blade stresses at 18,950 rpm.

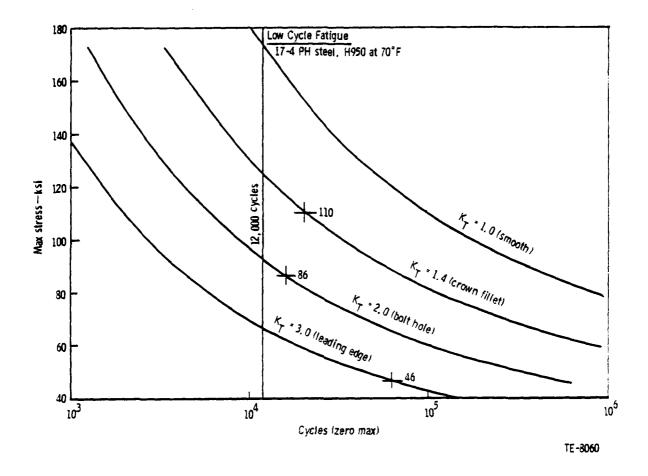
TABLE 2
Blade stress summary.

Type of Failure	<u>Criteria</u>	Allowable Stress and Location, ksi	Calcul Section Aver 1.8 R <sub>c</sub> , ksi 1	age Stress
Permanent set	95% F <sub>Ty</sub> @ 122% speed	137 crown	91.5	80.4
Burst	95% F <sub>Tu</sub> @ 130% speed	151 crown	103.9	91.3
			Calcula <u>Section Max</u> 1.8 R <sub>C</sub> , ksi l	Stress
Low cycle fatigue @ 100% speed	12,000 cycles K <sub>t</sub> = 3 K <sub>t</sub> = 3 K <sub>t</sub> = 1.4	66 lead edge 66 trial edge 125 crown	46.3 22.4 110.0	39.2 19.7 100.3
High cycle fatigue	+15 ksi vibratory 3 resonance +5 ksi vibratory 3 nonresonance	(Refer to Tab +5 ksi crown (required)	>le 3) +12.0 (allowable)	+14.4 (allowable)

necessary to scale the 61.5 ksi by the square of the speed ratio which gives an average stress level of 91.5 ksi at the overspeed condition. As shown in Table 2, this compares with an allowable stress of 137 ksi. Similarly, for a check of failure at 130% speed, the average stress scales to 104 ksi which compares with an allowable stress of 151 ksi.

Referring to the S-N diagram of Figure 25 at the 12,000-cycle requirement for low cycle fatigue, the airfoil leading and trailing edge allowable stresses are found to be 66 ksi based on a  $K_{\rm c}$  = 3.0. The crown fillet allowable stress is 125 ksi based on a  $K_{\rm c}$  = 1.4. Again, referring to Table 2, the calculated maximum principal stresses are well below these allowables.

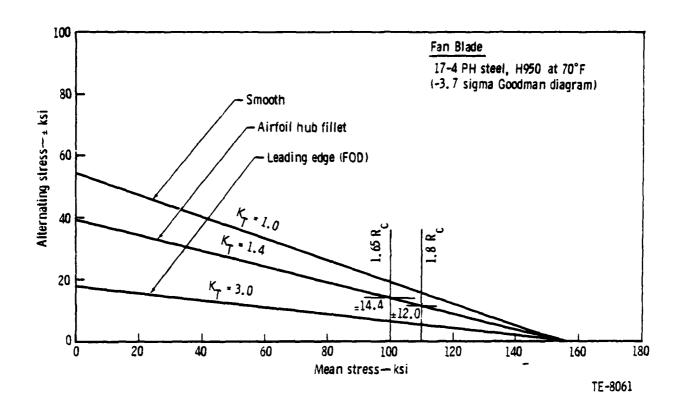
The Goodman diagram of Figure 26 indicates that the 110 ksi maximum steady stress at the hub fillet, in conjunction with a fillet radius concentration factor of  $K_{\rm t}$  = 1.4, provides a vibratory allowable stress at that location of  $\pm 12.0$  ksi. Since this is not a potential resonance condition, a level of  $\pm 5$  ksi would be considered satisfactory. The vibratory allowable stress at the hub fillet at the lower pressure ratio condition is  $\pm 14.4$  ksi.



P. B. S. C. L. S. C.

Figure 25. Fan blade and wheel S-N diagram.

Blade vibration analysis, to be discussed more fully later, indicates two potentially troublesome resonances in the operating envelope of the fan. These resonances are a second engine order-first bend mode coincidence at 11,800 rpm and a fourth engine order-first torsional mode coincidence at 18,800 rpm. Allowing a reasonable scatter of individual blade frequencies, the maximum static stresses at the maximum reasonance speed and at the critical vibratory stress points are given in Table 3. Entering the Goodman diagram of Figure 26 with those static stresses and the appropriate  $K_{\rm t}$  values, the allowable vibratory stresses shown in Table 3 are defined. All these allowables exceed the goal of  $\pm 15$  ksi vibratory allowable. It should also be noted that in the torsional mode, the dynamic stresses at location A and D are 50% and 85%, respectively, of the maximum dynamic stress which occurs at location B.



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Figure 26. Fan blade Goodman diagram.

TABLE 3
Steel blade stress summary at maximum dynamic response.

High cycle fatigue allowables		
17-4 PH steel	D	Suction
Cast properties	Pressure 5	

			Max Static		Allowable
		Resonance	Stress at		Vibratory
Mode	Location	Speed, rpm	Resonance, ksi	Kt	Stress, ksi
First bend	A	11,800	35.0	1.4	<u>+</u> 30.4
First torsion	A	18,800	95	1.4	<u>+</u> 15.7
	ם		23	3.0	<u>+</u> 15.6
	В		8	3.0	<u>+</u> 17.3

The airfoil, therefore, meets all static and dynamic stress criteria with appropriate K, factors in cast 17-4 PH material.

Wheel equivalent stresses are shown in Figure 27. Referring to Table 4, a web equivalent stress of 128 ksi at 122% of design speed compared with an allowable yield stress of 137 ksi assures no detrimental permanent set at that condition. Checking wheel burst at 130% of design speed finds a calculated mean hoop stress of 102 ksi and a maximum web radial stress of 127 ksi at that condition which compares with an allowable stress level of 137 ksi. Actual wheel burst is assumed to occur when the mean hoop stress of the wheel reaches 95% of the ultimate strength of the material. The burst speed thus calculated is 163% of design speed or 33,000 rpm. In terms of low cycle fatigue, the calculated values of stress at rim, web, and bore are all well under the allowable stresses taken from the S-N diagram of Figure 25 at 12,000 cycles and the appropriate K, factors.

TABLE 4
Steel wheel stress summary.

Type of Failure	<u> Criteria</u>	Allowable Stress and Location, ksi	Calculated stress, ksi	
			1.8 R <sub>c</sub>	1.65 R <sub>c</sub>
Permanent set	95% F <sub>T</sub> @ 122% speed	137 web equiv.	128	112.4
Burst	86% F <sub>T</sub> @ 130% speed	137 mean hoop	102	89.6
	-u	137 web radial	127	111.5
Low cycle fatigue	12,000 cycles @ 100% speed			
	$K_{T} = 1.4$	125 rim hoop	56	49
	$K_{T} = 2.0$ (bolt hole)	92.5 web equiv.	86	75.5
	K <sub>T</sub> = 1.0	174 bore hoop	92.8	81.5
	$K_{T} = 3.0$	67 balance holes	40	35

Therefore, the wheel also meets all design stress criteria in cast 17-4 PH material. The weight of the wheel and blades is approximately 16.3 lb.

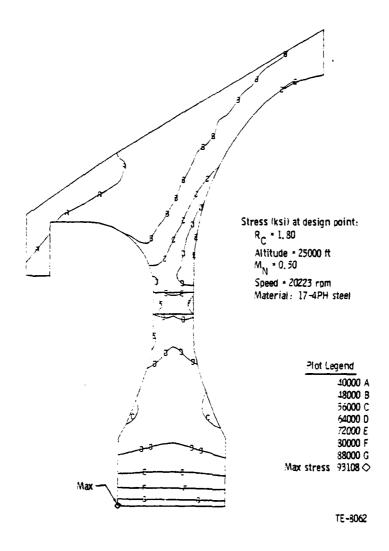


Figure 27. Wheel equivalent stresses.

# Titanium Rotor

Blade stresses for the titanium rotor are summarized in Table 5 where allowable stresses for permanent set and burst are comfortably above calculated stresses. In the area of low cycle fatigue however, calculated stresses exceed the allowable at the leading edge and equal the allowable stress at the trailing edge for the 1.8 pressure ratio condition. These stresses are calculated for a rigidly clamped airfoil; including wheel rim flexibility would lower the edge stresses significantly.

TABLE 5
Blade stress summary.

		Allowable Stress	Calc	ılated	
Type of Failure	Criteria	and Location, ksi	Section Ave	erage Stress	
			1.8 R <sub>c</sub> , ksi	1.65 R <sub>c</sub> , ksí	
Permanent set	95% F <sub>T</sub> 3 122% speed	92 crown	57.5	50.5	
Bursc	95Z F <sub>T</sub> y @ 130Z speed	99 crown	65.3	57.4	
			Calc	ılated	
			Section :	fax Stress	
		•	1.8 R <sub>c</sub> , ksi	1.65 R <sub>c</sub> , k <b>s</b> i	
Low cycle fatigue	12,000 cycles K <sub>E</sub> = 3	39 lead edge	50	<u>ئە</u>	
9 100% speed	K ू = 3	39 trail edge	39	34.5	
	K_ = 1.4	77 crown	53	47	
High cycle fatigue	+15 ksi vibratory @ resonance	(Re	fer to Table 3)		
	+5 ksi vibratory @ nonresonance	±5 ksi crown	10.0	11.1	
		(required)	(allowable)	(allowable)	

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Referring to Table 6, there are two resonances in the operating envelope of the titanium fan. The second engine order-first bend mode coincidence at 11,500 rpm produces a maximum vibratory response at location A where the static stress is 11.6 KSI. With a fillet radius concentration factor of 1.4 the allowable stress in cast titanium is ±17.5 KSI which exceeds the requirement of ±15 KSI. Similarly, the fourth engine order-first torsional mode coincidence at 18,800 rpm produces an acceptable allowable dynamic stress of ±18.4 KSI.

The titanium wheel stresses are summarized in Table 7 where the allowable stresses are seen to exceed calculated stresses by comfortable margins for all conditions. The calculated burst speed of the titanium wheel is 34,500 rpm or 171% of design speed.

The weight of the titanium wheel and blades is approximately 9.3 lbs, or approximately 7 lbs less than the steel rotor.

TABLE 6
Titanium blade stress summary at maximum dynamic response.

High cycle fatigue allowables Titanium-6-4 Cast

Mode	Location	Resonance Speed, rpm	Max Static Stress at Resonance, ksi	<u>K</u> t	Allowable Vibratory Stress, ksi
First bend First torsion	A	11,500	11.6	1.4	<u>+</u> 17.5
	B	18,800	7.1	1.4	<u>+</u> 18.4

TABLE 7
Titanium wheel stress summary.

Type of Failure	Criteria	Allowable Stress Criteria and Location, ksi		Calculated stress, ksi	
			1.8R <sub>c</sub>	1.65 R <sub>c</sub>	
Permanent set	95% F <sub>T</sub> @ 122% speed	92 web equiv.	79	69	
Burst	86% F <sub>T.</sub> @ 130% speed	89 mean hoop	56.7	49.8	
	-u	89 web radial	79	69	
Low cycle fatigue	12,000 cycles @ 100% speed				
	$K_{\rm p} = 1.4$	77 rim hoop	32	28	
	$\kappa_{\mathrm{T}}^{2} = 2.0$	56 web equiv.	53	47	
	$K_{T} = 1.0$	102 bore hoop	52	46	

# VIBRATION ANALYSIS

Dynamic analyses of the airfoil, both vibration and flutter, are unaffected by the material change.

Frequencies, mode shapes, and relative dynamic stress distributions for all modes up through the vane passage frequency (42 EO) were calculated, using finite element techniques. The frequency versus speed interference diagram showing the first three modes is shown in Figure 28. The overall interference diagram is presented in Figure 29. Note that the first bending mode (1B) crosses 2 EO at relatively low speed (60%) such that the excitation levels due to inlet distortion will be low.

The relative dynamic stress distributions are determined to locate the maximum dynamic stress location for each mode to assess, in combination with the steady-state stress calculation, the allowable vibratory stress levels. Figures 30 and 31 show the relative radial dynamic stress distributions for the first two modes. These first two modes are given particular emphasis since the excitation force levels produced by the lower four engine orders are us-

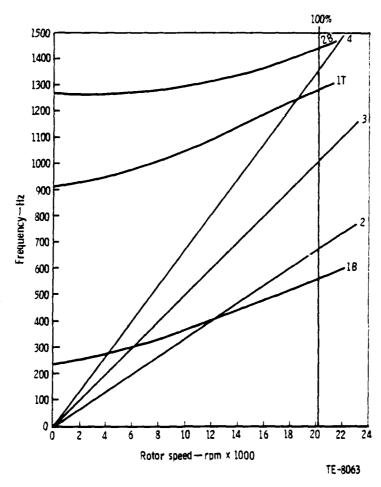


Figure 28. Frequency-speed interference diagram (first 3 modes).

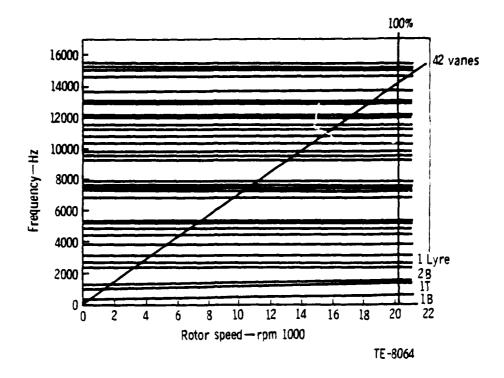


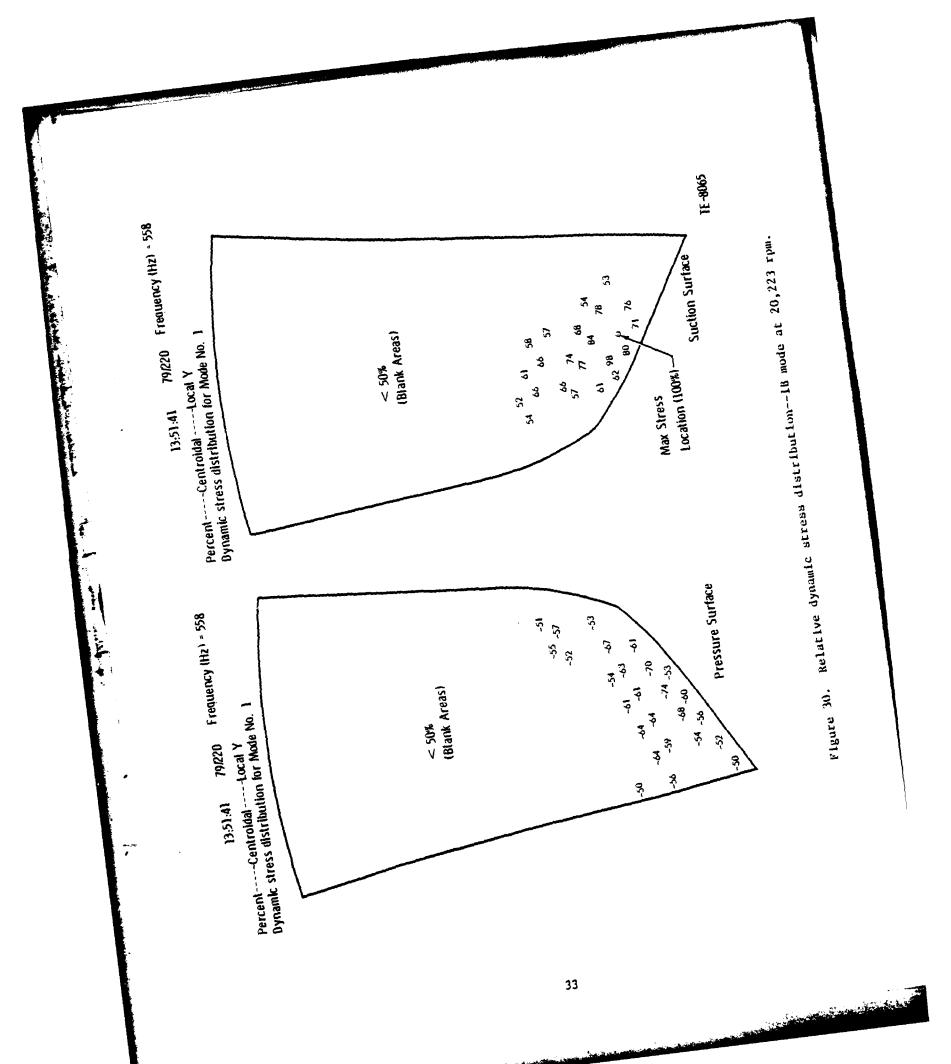
Figure 29. Frequency-speed interference diagram (first 32 modes).

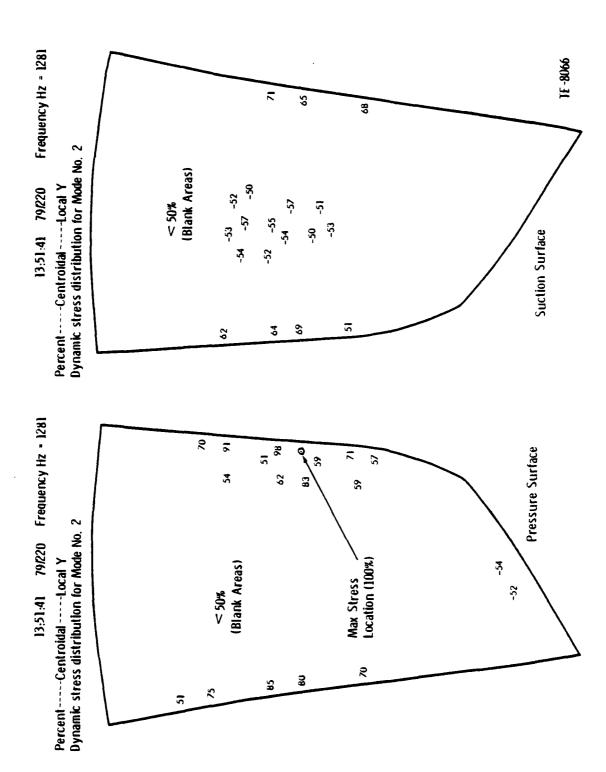
ually higher than those generated by higher harmonics of rotor rotation. As previously discussed, the results of the allowable vibratory stress determination using the Goodman diagram, predicted steady-state stress at coincidence speed, and relative dynamic stress distributions satisfied the ±15 ksi vibratory stress criteria.

Blade response because of a coincidence of high modes with vane passage (42 EO) are expected to be very low because of the large axial distance (1.5 chord lengths) that the vane row is located aft of the blade row. This large spacing is a result of the noise design criteria.

# FLUTTER ANALYSIS

The results of the torsional stall flutter analysis are shown in Figure 32. The predicted margin of safety is an adequate 4 deg of incidence angle above the estimated operating line. The calculated bending stall flutter reduced frequency parameter is well above the 0.25 criterion at 0.302. Similarily, the supersonic unstalled reduced torsional frequency parameter at 0.68 satisfies the 0.60 requirement.





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Relative dynamic stress distribution--1T mode at 20,223 rpm. Figure 31.

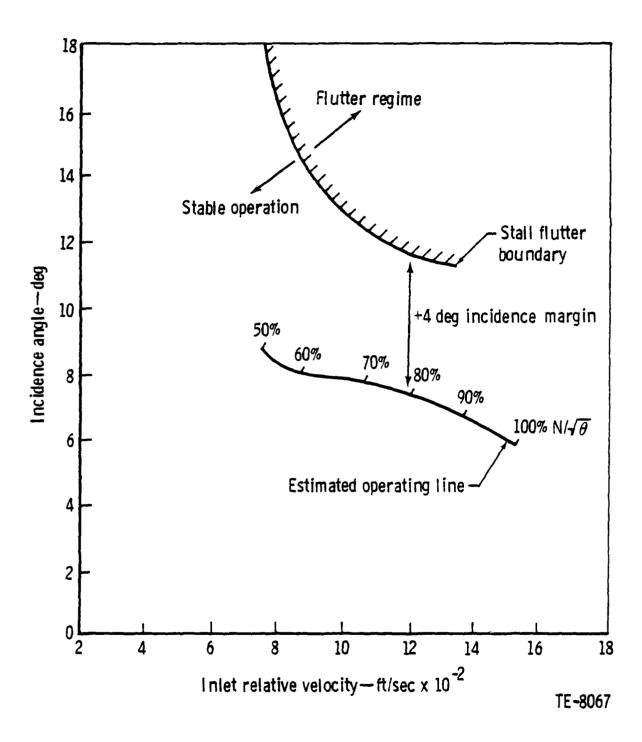


Figure 32. Stall torsional flutter analysis.

#### BIRD INGESTION ANALYSIS

The USAF requirements for bird ingestion are defined in Mil-E-5007D, paragraph 3.2.5.6.1. A summary of the specified bird sizes and engine conditions is given in Table 8. The small high bypass fan and typical trainer aircraft speeds corresponding to the Mil-E-5007D requirements are given in Table 9. In Table 9, the aircraft liftoff, climb, cruise, and descent speeds for the 1.80 pressure ratio fan are assumed to be the same as for the 1.65 pressure ratio fan application.

The annular inlet area of this fan is less than 200 in. which sets the maximum bird size at two pounds (Ref. Mil-E-5007D par 3.2.5.6.le). For bird impacts up to 2 lb, no failure shall result which will cause shutdown of the engine although some damage to engine parts may occur.

The failure mode considered here for bird ingestion is local impact damage in the leading edge region of the blade. The calculation of a local damage index, based on a Lycoming criterion approach (Ref. FAA-RD-77-55), has been incorporated into the DDA bird ingestion analysis. This approach relates significant bird slice, impact area, and airfoil parameters to a damage index value that corresponds to critical blade damage. An acceptable damage index level is determined by correlation with actual bird ingestion test data. Engine survivability for new blade designs at the critical ingestion conditions is then predicted with some confidence by use of the damage index calculation.

The shear-penetration damage index is expressed as

$$v_{i} = 0.273 \text{ V}_{N} \sqrt{\frac{\pi/4 \left(\frac{eD_{B}R_{B}}{h\tau_{y}}\right)}{}}$$

## where:

- $v_N$  = the normal component of impact velocity (ft/sec)
- e = the bird density = 0.045 lb/in.<sup>3</sup>
- D<sub>R</sub> = the bird diameter (in.)
- h = the target mid-thickness (in.)
- $\tau_{y}$  = the target material shear yield (psi)
- K<sub>R</sub> = the bird fragmentation parameter

## TABLE 8

# Mil-E-5007D bird ingestion requirements.

- A. Birds weighing 2 to 4 ounces (a maximum of sixteen at a time) and birds weighing 2 pounds (one at a time) ingested at a bird velocity equal to the take-off flight speed, with the engine at maximum rated speed.
- B. Birds weighing 2 to 4 ounces (a maximum of sixteen at a time) and birds weighing 2 pounds (one at a time) ingested at a bird velocity equal to the cruise flight speed with the engine at maximum continuous speed.
- C. Birds weighing 2 to 4 ounces (a maximum of sixteen at a time) and birds weighing 2 pounds (one at a time) ingested at a bird velocity equal to the descent flight speed with the engine at an associated engine speed.
- D. Birds weighing 4 pounds ingested at a bird velocity based on the most critical flight speed with the engine at maximum rated speed.

Note: Condition D does not apply since the trainer fan inlet is less than 200 in<sup>2</sup>. A maximum of four birds weighing 2 to 4 ounces must be considered for the trainer fan.

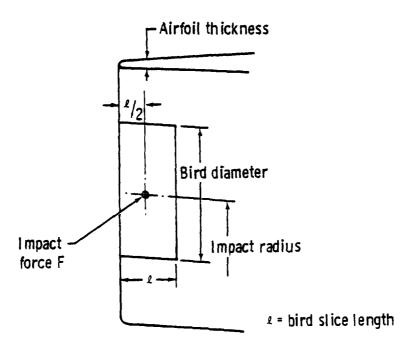
TABLE 9
Fan and aircraft speeds for Mil-E-5007D conditions.

	Pressure ratio	
	1.65	1.80
Condition A		
Engine take-off fan speed (rpm)	17,303	18,465
Lift-off speed of typical aircraft (kts)	90	90
Climb speed of typical aircraft (kts)	198	198
Condition B		
Engine maximum continuous fan speed (rpm)	17,292	18,454
Cruise speed of typical aircraft (kts)	194	1 94
Condition C		
Engine descent fan speed (rpm)	10,377	11,074
Descent speed of typical aircraft (kts)	198	198
Condition D		
Engine cruise fan speed (rpm)	10,496	11,201
Cruise speed of typical aircraft (kts)	194	194

The values of  $\mathbf{D}_{\mathbf{B}}$  and  $\mathbf{K}_{\mathbf{B}}$  are defined by:

$$R_{B} = \begin{cases} 1.0 & V_{N} \le 260 \text{ft/sec} \\ 2.40-0.0054 V_{N} & \text{for} \\ 2.5 & 400 \le V_{N} \end{cases}$$

Figure 33 illustrates the region of leading edge impact.

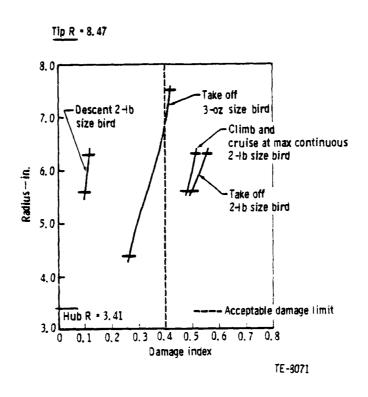


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Figure 33. Bird impact area and blade thickness illustration.

The ballistic-limit velocity for total shear penetration is reached as the damage index approaches 1.0 and theoretically should cause maximum structural damage. In practice, the maximum allowable damage index must be determined by correlation with test.

The calculated damage index along the airfold span for a base-line titanium design at a 1.80 pressure ratio is shown in Figure 34. The maximum damage is predicted to occur at the outermost radial position permitted by the outer case and the 2-lb bird diameter. The maximum allowable damage index for shroudless airfoils is set at 0.40 which limits the expected damage to spanwise tears. A damage index greater than 0.40 could produce loss of airfoil section (for several blades at the 2-lb size) and would cause rotor unbalance that would require engine shut-down. On this basis, the base-line titanium design is judged unacceptable.



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Figure 34. Bird ingestion damage index--titanium blade.

A design study was made to establish the damage index sensitivity to leading edge radius, thickness/chord, number of blades, etc, to identify an acceptable damage index domain for the titanium fan. A titanium airfoil with a 0.025-in. leading edge radius was found to have an acceptable index. However, the performance penalties associated with the required design changes were excessive. Thus, based on the bird ingestion requirements (which are particularly important for trainer engines), a switch to 17-4 PH stainless material was made. The advantage of the material is an increase in shear strength from 66.5 to 106 ksi.

The calculated damage index along the airfoil span for the steel design is shown in Figure 35. Again, the maximum index value of 0.39 is found to occur at the outermost radial position permitted by the outer case for the 2-lb size bird. The steel design (with a 0.0125 leading edge radius) thus satisfies the acceptable damage index limit of 0.40 for the this type of blading.

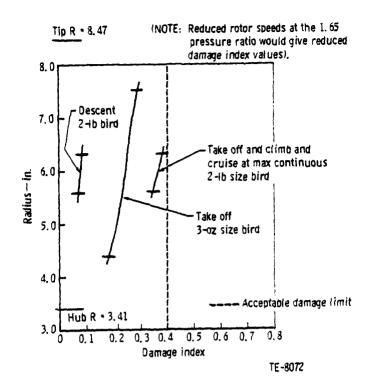


Figure 35. Bird ingestion damage index-steel blade.

Bird ingestion requirements have forced a change in leading edge thickness and material for this fan blade. The penalties are approximately one-half percentage point in stage efficiency and an increase in rotor weight from 9.25 to 16.25 lb. Any degradation in engine response during power transients have not been quantified.

# SECTION V NOISE PREDICTION

Noise goals for the high bypass turbofan powered undergraduate trainer are to be in compliance with Federal Air Regulation Part 36 requirements and maintain ground idle noise substantially below current levels. Engine cycle and fan design data were combined to estimate trainer noise levels for Part 36 and idle conditions. A brief description of the noise prediction methods used and the noise levels estimated for the undergraduate trainer are presented in this section.

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Estimated noise levels for a PD 418 powered trainer are presented in Table 10 and show that the above noise goals are met.

TABLE 10
Undergraduate trainer noise levels.

# FAR Part 36 Levels (EPNdB)

	Takeoff	Approach
Part 36 Requirements	89	98
PD418 Powered Trainer	78	93

Ground Idle Tone Corrected Perceived Noise Levels

# PNdBt at 250-ft Radius (Single Engine)

	Front	Rear	
T37 with J69-T-25	118.3	99	
PD418 Powered Trainer	79	84	

Noise estimates were made through a DDA computer program developed for turbofan noise prediction. A noise generation model for each source, fan, jet, turbine, and combustor, is contained in the program so that the noise output from each source is dependent on its individual operating conditions and engine total noise reflects the contribution from each noise source.

High bypass ratio turbofan engines are usually fan noise dominated with the jet and combustor being secondary sources. The PD418 engine incorporates the following design features to reduce fan and jet noise generation:

- o Ample space between the fan and the outlet guide vanes (1.5 fan chords)
- o Ratio of outlet guide vanes to fan blades > 2 to cut off blade passing tone
- o Internal mixer to reduce nozzle exit velocity

The noise estimates for the PD418 include the noise reductions provided by these features. The PD418 incorporates a 1.8:1 pressure ratio fan (design point) matched to the GMA500 core and operates at part speed for takeoff, climb, and approach. The engine is fan noise dominated at these conditions so that fan duct acoustic treatment could be used to achieve levels lower than the 78 and 93 EPNdB predicted for takeoff and approach. These levels are 11 and 5 EPNdB below the FAR Part 36 requirements.

At the ground idle condition, the PD418 peak levels at 250-ft radius are expected to be 79 and 84 PNdBt (tone corrected PNdB) in front and rear of the aircraft. In dBA units, front and rear levels are 60 and 67. These levels translate into noise reductions of 34 PNdBt or 35 dBA when the PD418 trainer is compared with the T37B with the J69-T-25 engine.\* In predicting the ground idle noise levels, it was assumed that ingested ground level turbulence would prevent cutoff and so the blade passing tone is included in the above levels.

<sup>\*</sup>Speakman, J. D., Power R. G., and Lee, R. A., Community Noise Exposure
Resulting from Aircraft Operations, AMRL-TR-73-110, Vol. 4, February 1978.

## APPENDIX A

## AXIAL COMPRESSOR DESIGN SYSTEM

The vector diagram calculation used for axial compressor design assumes an axisymmetric flow field and obtains a solution of the continuity, energy, and radial equilibrium equations. The design analysis is identified as the Axial Compressor Design System (ACDS) Program BD76. Viscous terms are omitted; however, the equations do account for streamline curvature, radial gradients of total enthalpy and entropy, and blade force terms arising from non-radial blade surfaces. Calculations may be performed at the leading or trailing edges of the airfoils by slanting the calculation stations.

Enthalpy rise across a rotor is given by Euler's turbine equation, and the continuity equation is adjusted for local as well as endwall blockage.

Used as a design tool, the calculation provides detailed examination of the aerothermodynamic solution of the flow process through the compressor. The solution is iterative and must rely on profile loss estimates which are correlated as a function of aerodynamic loading (diffusion factor). This data has been obtained from test data for a wide range of compressor designs and is continually updated.

The equilibrium equation is in the form of:

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$$\frac{dV_{z}^{2}}{dr} = -\frac{d(V_{\theta}^{2})}{dr} = -\frac{d(V_{r}^{2})}{dr} = -\frac{d(V_{r}^{2})}{dr} = +2 \left[ \frac{(dH_{0})}{dr} - T \frac{ds}{dr} \right]_{c}$$

$$+2 \left[ V_{z} \frac{dV_{r}}{dz} - \frac{V_{\theta}^{2}}{r} \right] + 2V_{z} \frac{dV_{z}}{dz} = -\frac{dV_{z}}{dr} = +$$

$$2V_{z} \frac{d(rV_{\theta})}{dz} = \frac{d\theta}{dr} = -\frac{d\theta}{dr} = -\frac{d\theta}{dr}$$

## where:

r radial distance

z axial distance

 $\theta$  tangential distance

V\_ radial velocity

V axial velocity

V<sub>Q</sub> tangential veolicty

T total temperature

s entropy

HA total enthalpy

c projection of the calculating station on relative stress surface

relative to stream surface

The continuity equation is:

$$W_a = 2\pi \int_{y_h}^{y_t} K_{\gamma} \rho V_{m} \sin (\lambda - \epsilon) rdy$$

# where:

W<sub>a</sub> airflow

V\_ meridional velocity

K<sub>γ</sub> blockage factor

ρ density

Y length along the calculating station

angle between tangent to the streamline projected on the

meridional plan and axial direction

 $\lambda$  angle between calculation station and axial

APPENDIX B

DESIGN POINT VECTOR DIAGRAMS

TABLE B-1. Velocity diagrams.

ROTOR INLET

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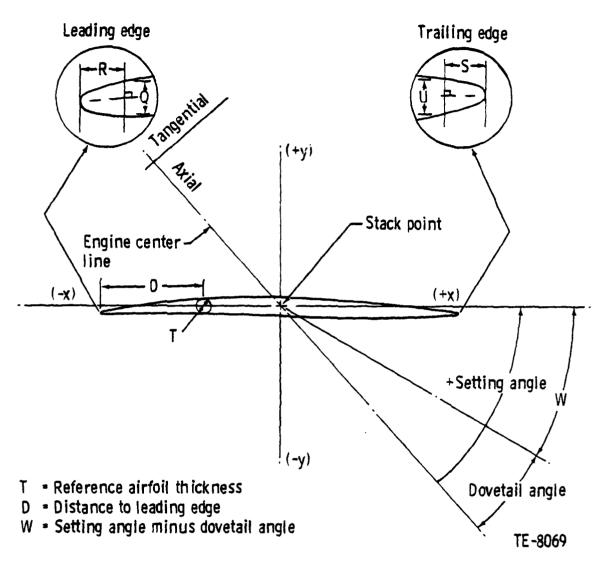
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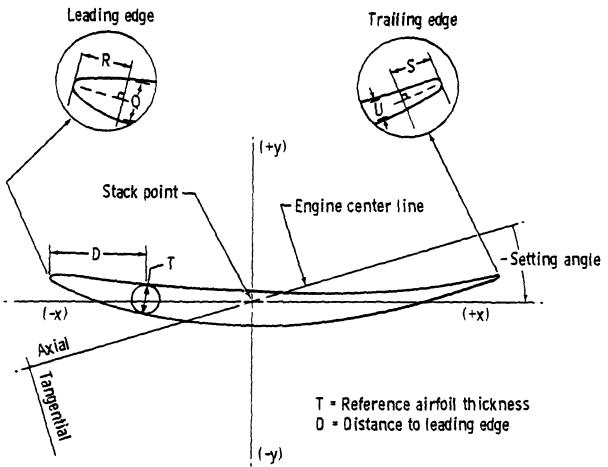
APPENDIX C

ROTOR AND STATOR BLADE COORDINATES



Rotation is counterclockwise from the rear

Figure C-1. Blade manufacturing dimension definitions.



Rotation is counterclockwise from the rear

TE-8070

Figure C-2. Vane manufacturing dimension definitions.

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                                       FAN COMPRESSOR VANE
                                           REFERENCE DISTANCE ANDIAL SETTING AIRFOIL TO LEADING DISTANCE ANGLE THICKNESS EDGE
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                       LEADING EDGE AXIAL TANGENT POINT -0.5719
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S DIMENSION 0.0600
                        STACK POINT COURDINATES 0.0 . 0.0 . C.C. CENTER OF GRAVITY COORDINATES -0.0469, -0.0171 COMPRESSOR ROTATION IS COUNTER CLUCKWISE FROM THE REAR
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STATION

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2-0.3518 0.1505 22 0.4079 0.0576 42 0.3218 -0.3319
3-0.5138 0.1520 23 0.4079 0.0576 42 0.3218 -0.3319
3-0.4127 0.1205 23 0.4093 0.30766 43 0.2752 -0.0375
6-0.4088 0.1205 20 0.5043 0.30766 45 0.1202 -0.0413
5-0.4127 0.1205 20 0.5043 0.30766 45 0.1202 -0.0413
6-0.3478 0.1205 20 0.5048 0.1001 40 0.1218 -0.0349
7-0.3227 0.1105 27 0.4088 0.1001 40 0.1218 -0.0349
8-0.3228 0.1004 20 0.4087 0.1180 47 0.0634 -0.0349
9-0.2220 0.0924 29 0.7317 0.1502 49 -0.0291 -0.0212
10-0.1772 0.0929 30 0.7417 0.1502 49 -0.0291 -0.0212
11-0.1112 0.0761 31 0.7498 0.1508 51 -0.1324 -0.0014
12-0.4784 0.0697 32 0.7477 0.1513 51 -0.1324 -0.0014
12-0.4784 0.0697 32 0.7477 0.1513 51 -0.2347 0.02212
13-0.0244 0.0697 32 0.7477 0.1515 54 -0.2798 0.0374
15 0.0244 0.0623 33 0.7437 0.1446 53 -0.2347 0.0224
10 0.1246 0.0623 33 0.7437 0.1446 53 -0.2347 0.0224
10 0.1246 0.0623 33 0.7437 0.1446 53 -0.2347 0.0014
17 0.1034 0.0692 34 0.0693 0.0678
19 0.0244 0.0623 35 0.0634 0.0693 0.0678
10 0.1246 0.0637 32 0.0634 0.0693 0.0678
10 0.1246 0.0636 0.0637 0.06374
10 0.0248 0.0698 0.0698 0.0698 0.0679 0.07540
11 0.1034 0.0648 0.0649 0.0659 0.06780 0.06780
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. INDICATES EXTREME POINTS

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CSC 12122

DISTANCE TO LEADING EDGE MEFERENCE AIRFUIL THICKNESS SETTING ANGLE RADII JACIAL JISTANCE 0.006 6.5336 -ZT.296 5.6933 -210 17M +6\$ 0.7450

LEAUING EDGE AAIAL TANGENT PUINT -0.3921

1 UNENSION 0.0264 U UNENSION 0.0260 R DIMENSION 0.0600 S ORMENSION 0.0600 D DEMENSION 0.0600

# REFERENCE COORDINATE POINTS

NG. X			•	•		•	' -
1 -0.4874* -0.4779 3 -0.4309 4 -0.3254 5 -0.3254 6 -0.2317 7 -0.1732 10 -0.0798 11 -0.0254 12 0.0836 13 0.1772 16 0.2896 17 0.2896 18 0.3283 19 0.3283	0-1998 0-1907 0-1967 0-17-99 0-15718 0	124567340LUR 4567398841114067340733737373737373737373737373737373737	0.4857 0.5310 0.5710 0.6710 0.6710 0.6710 0.3710 0.381	0.112925 0.112925 0.112925 0.112925 0.112925 0.112929 0.112929 0.112929 0.11297794 0.11297794 0.11297794 0.11297794 0.11297794	12345678901234567890	0.5000 0.3908 0.3908 0.32792 0.27902 0.1093 0.07313 0.07313 -0.01343 -0.07313 -0.07313 -0.1874 -0.1874 -0.1874 -0.1874 -0.4884	0.0151 0.0047 -0.0008 -0.0042 -0.0047 0.0110 0.0185 0.0293 0.0293 0.0393 0.0791 0.0134 0.1304 0.1334 0.1723 0.1723

. INDICATES EXTREME POINTS

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PAGE 11

FAN LOM	PRESSCH VANE		<b>LSC 12122</b>		
RAUIAL DISTANCE	Se ITING ANGLE	REFERENCE AIRFOIL THICKNESS	DISTANCE TO LEADING EDGE	L.E.	
6.5007	-20 917 -200 544 375	C.1012	J.7450	0.008	7.004

CSC 12122

LÉAUING CLUC AXIAL TANGENT POINT -0.2390

O DIMENSION 0.0270 U JUNENSION 0.0246

STACK POINT COURDINATES 0.0 . 0.0 . CENTER OF GRAVITY COURDINATES 0.0.00 . COLUMN THE REAR COMPRESSUM ROTATION IS COUNTER CLUCKHISE FROM THE REAR

# REFERENCE COORDINATE POINTS

STATION X	*		*	4		×	<b>Y</b>
1 -0.44 -0.33 -0.33 -0.23 -0.23 -0.23 -0.20 -0.20 -0.20 -0.20	932 0.2508 932 0.2313 935 0.2110 937 0.4125 774 0.4125 777 0.1727 207 0.1635 209 0.1635 960 0.1637 970 0.1633 970 0.1633	123456797012345678	911172594 655272594 6577873594 6577873594 677873594 677873594 67787594 67787594 67787594 67787594 67787594 67787594	7401563611109451164 3450475353837444537645115 1.111124444337645115 1.11112444433765115 1.11112444433765 1.11112444433765 1.11112444433765 1.11112444433765 1.1111244433765 1.1111244433765 1.111124433765 1.11112443	143450739012345678	298537 V9 D2 29537 9 6953	05104 05104 0033243 00033243 00000 00000 00000 00000 00000 00000 0000
13 0-14 1- 0-2 15 0-3 17 0-3	900 0.1539 391 0.1-75 876 0.1-23 476 0.1359	34 35 36	1.6201 0.9696 0.9275	0.1945 0.1091 0.1391 0.1166	53 55 56 57 58 60	-0.3917 -0.1519 -0.1996 -0.2469	

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FAN LUMPRESSUR VANE
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                                                                                                                                                                                                                                                                                                                                                                       REFERENCE
AIRFOIL
THICKNESS
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TO LEADING
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                                                       Q JIMENSION 0.0276
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                                                                                                                               REFERENCE COORDINATE POINTS
STATION 4
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                               . INDICATES EXTREME POINTS
             TO AUG 79
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                 PAGE 19
                                                                                                                                                                                                                                                                                                          CSC 12122
                                    FAN COMPRESSUR VANE
                                     REFERENCE DISTANCE
AIRFUIL TO LEADING
THICKNESS EDGE
                                                                                                                                                                                                                                                                                                                                                                                                                                                       RADII
                           RADIAL
DISTANCE
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