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COOLING ACOUSTIC TRANSDUCER WITH HEAT PIPES

STATEMENT OF GOVERNMENT INTEREST

[0001] The invention described herein may be manufactured and used by or for the Government of the United States of America for governmental purposes without the payment of any royalties thereon or therefor.

CROSS REFERENCE TO OTHER PATENT APPLICATIONS

[0002] None.

BACKGROUND OF THE INVENTION

(1) Field of the Invention

[0003] The present invention utilizes heat pipe technology to cool the ceramic used in transducers and acoustic projectors.

(2) Description of the Prior Art

[0004] It is known in the art that transducers, designed to project acoustic power, are often limited by the build-up of internal heat generated in an active piezoelectric ceramic as a result of dielectric losses. High internal temperatures can adversely change material properties, increasing losses, possibly causing de-poling, and reducing reliability and performance.

A variety of cooling methods are used to address the heat dissipation problem.

[0005] In Cluzel et al. (United States Patent No. 4,031,418), a piezo-electric transducer for low frequency acoustical waves comprising a stack of piezo-electric elements and alternating electrodes disposed between a front receiver plate and a counter mass. The counter mass comprises a rigid annular block surrounding the stack, a rear plate engaging the rear of the stack and an elastic connection between the block and the rear plate.

[0006] In Snyder (United States Patent No. 5,721,463), a device is disclosed for improving thermal transfer inside an ultrasound probe and reducing heat build-up near the transducer face. The cable components are used as heat conductors which conduct heat out of the probe handle. They are coupled in an internal heat conductor which is in a heat conductive relationship with the transducer pallet. Thus, heat generated by the transducer array can be transferred, via the heat conductor plate and the cable heat conductors, away from the probe surface. A heat conductive structure can be embedded in the overall shield braid of the cable. Suitable heat conductive structures include thread or wire made of material having a high coefficient of thermal conductivity, as well as narrow tubing filled with heat conductive fluid. Alternatively, inlet and return flow paths for cooling fluid are incorporated in the cable. The inlet and

return flow paths inside the cable are respectively connected to the inlet and outlet of a flow path which is in heat conductive relationship with an internal heat conductor in the probe handle.

[0007] In Austin et al. (United States Patent No. 5,884,693), a passive cooling system is disclosed for cooling an enclosure containing electronic components. A hollowed portion of the enclosure is formed as an integral heat pipe containing a working fluid. The hollowed portion has an evaporator section located at the top and a condenser section located at the bottom. The enclosure also has hollowed side walls which serve as passageways for the working fluid to flow through in between the evaporator and condenser sections. Gravity and the pressure of evaporation force the working fluid down to the condenser section. A wick is provided for returning the working fluid to the evaporator section by capillary action. Additionally, an ultrasonic transducer driven by the heat rejected from the condenser section may be used to help return the working fluid to the evaporator section. Finally, a check valve may be employed before the evaporator section for the working fluid to fluid.

[0008] In Kelly, Jr. et al (United States Patent No. 5,961,465), an ultrasound transducer structure is disclosed which includes: an ultrasound transducer operable to generate and receive ultrasonic energy, a communication cable, integrated circuits for processing signals received from said ultrasound transducer and flexible circuits for connecting the communication

cable to the integrated circuits to the ultrasound transducer. A housing contains the ultrasound transducer, the integrated circuits and the flexible circuits. A heat transfer structure is positioned within the housing and is in contact with the integrated circuit. A heat conductor resides in contact with the heat transfer structure and conducts heat generated by the integrated circuits to a heat sink.

[0009] In Kan et al (United States Patent No. 6,528,909), a spindle motor assembly is disclosed which has a shaft with an integral heat pipe. The shaft with the integral heat pipe improves the thermal conductivity of the shaft and the spindle motor assembly. The shaft includes an elongated portion and a sealing structure. For one embodiment, the sealing structure includes a cap and a gasket that are joined to the shaft by a brazing process.

[0010] Baumgartner et al (United States Patent No. 7,017,245), a method is disclosed for manufacturing a multi-layer acoustic transducer with reduced total electrical impedance. The method is based on the bonding of two piezoelectric ceramic layers with confronting metalized surfaces to a thin electrical conductor, then electrically connecting the top and bottom surfaces to form a wrap-around electrode while a center conductor forms a second electrode. The total electrical impedance of a two-layer ceramic stack comprised of piezo-electric layers connected in this manner is one-fourth that of a solid ceramic layer of the same size.

This provides for better matching of the acoustic stack impedance to that of the electrical cable, increased penetration depth for imaging within the body, and improved acoustic element sensitivity.

[0011] In regard to the references above, heat pipes having a wick are taught for a metal shaft in an electric motor (United States Patent No. 6,528,909), which could be construed as substantially equivalent to a hollow bolt in the Tonpilz design (a hollow bolt may have synergistic benefits and this would not be the case for a shaft). United States Patent No. 5,884,693 discloses a passive cooling system for cooling an enclosure containing electronic components; however, no combination of the cited references suggests or teaches all of the elements of the acoustic transducer cooled with heat pipes in such a manner that would be predictable to one skilled in the art.

[0012] Such a heat pipe would be a closed tube with a working fluid that vaporizes at the hot end and condenses at the cold end; thereby, transferring large amounts of heat by removing the latent heat of vaporization at the hot end and adding the heat at the cold end. Furthermore, no transducer in any of the transducers of the prior art contains off-center heat pipes.

[0013] Most transducer packages involve a stack of active ceramic. A Tonpilz transducer 10 in the prior art, as depicted in FIG. 1, consists of a stack of ring elements 12. Pre-stressed

by a center bolt 14 with a radiating piston 16 on one end (the radiating piston is in contact with the surrounding water and radiates acoustic energy into the water by vibrating at acoustic frequencies). The other end of the radiating piston 16 is connected to a tail mass 18. The tail mass 18 governs the resonant frequency, which is $(k/m)^{1/2}$ (where k and m are the effective stiffness and mass, respectively, of the transducer) and is heavy compared to the radiating piston 16 so that the tail mass reduces recoil as a result of the motion of the radiating piston.

[0014] Typical attempts at thermal management involve heat-sinking the stack ends to the highly thermally-conductive piston 16 and end mass (e.g., constructed from aluminum). The piston 16 is in contact with (relatively cool) surrounding fluid.

[0015] The center bolt 14 pre-stresses the ceramic in order to prevent the ceramic from going into tension during operation; otherwise, the tension would cause the ceramic to break. The stiffness of the center bolt 14 must be small compared to the stack stiffness to avoid restraining the motion of the end masses and increasing the resonant frequency.

[0016] Another common design for transducer cooling is a flex-tensional transducer 20, shown in the prior art of FIG. 2. In the figure, a ceramic stack 22 drives a shell 24; thereby, leading to larger displacement of the surrounding fluid because

of the shell geometry. The ceramic stack 22 expands and contracts because of the piezo-electric effect. This expansion of the ceramic stack 22 causes the shell to expand and contract against the surrounding fluid with the result of radiating acoustic energy. The shell 24 also keeps the ceramic stack 22 in compression.

SUMMARY OF THE INVENTION

[0017] It is therefore a general purpose and primary object of the present invention to provide heat pipe technology to cool the ceramic used in acoustic projectors.

[0018] To attain the object of the present invention, a heat pipe is provided as a closed tube with a lower end and an upper end that are respectively in contact with a hot surface and a cold surface. The hot surface is in contact with the interior of the transducer, and the cold surface is in contact with the surrounding water, or a cooler part of the transducer. A working fluid is used in which the fluid will boil at the temperature of the hot surface and condense at the temperature of the cold surface.

[0019] A wick on the inside of the heat pipe facilitates the return of the condensed fluid to the hot end. The wick is primarily needed when the fluid must return by capillary action

against gravity. The heat pipe can be evacuated to adjust the boiling temperature of the working fluid and, by extension, the effective temperature range.

[0020] A variant of the present invention involves drilling additional holes into ceramic rings of the piezoceramic stack and inserting heat pipes. The heat pipes can be comparatively small in diameter compared to the diameter of the stack. Furthermore, increasing the length of the heat pipe into the tail mass and the piston also increases the length of the "cool" region for the working fluid to condense; thereby improving the performance of the heat pipe.

[0021] A further variant of the present invention (involving flex tensional transducers) involves a heat pipe either in the center or in multiple locations in the ceramic rings to conduct heat to the shell. This use of heat pipes in the flex tensional transducer has the advantage that both ends of the heat pipe are thermally "shorted" to the surrounding fluid, but the disadvantage of a shorter "cool" region for the heat pipe.

BRIEF DESCRIPTION OF THE DRAWINGS

[0022] A more complete understanding of the invention and many of the attendant advantages thereto will be readily appreciated as the same becomes better understood by reference to the following detailed description when considered in conjunction

with the accompanying drawings wherein like reference numerals and symbols designate identical or corresponding parts throughout the several views and wherein:

[0023] FIG. 1 is a prior art depiction of a Tonpilz transducer design;

[0024] FIG. 2 depicts a prior art flex-tensional transducer design;

[0025] FIG. 3 depicts a heat pipe of the present invention;

[0026] FIG. 4 depicts an embodiment of transducer cooling using the heat pipe of the present invention; and

[0027] FIG. 5 depicts a first variant of the embodiment of the present invention.

DETAILED DESCRIPTION OF THE INVENTION

[0028] The slender ceramic stack geometry and thermal profile of a typical transducer make the transducer ideally suited to cooling by means of heat pipes. Heat pipes use changes of phase to achieve a very high rate of heat transfer.

[0029] A heat pipe 30 of the present invention is shown in FIG. 3. In the figure, a closed tube 10 is provided whose lower end 12 and an upper end 14 are respectively in contact with a hot surface and a cold surface. The hot surface is in contact with the interior of the transducer, and the cold surface is in

contact with the surrounding water, or a cooler part of the transducer (closer to the surrounding water). A working fluid 100 (such as ammonia, alcohol, ethanol, water or some combination thereof) is used in which the fluid will boil at the temperature of the hot surface and condense at the temperature of the cold surface - either at atmospheric pressure or in a partial vacuum.

[0030] A wick 32 on the inside of the heat pipe 30 facilitates the return of the condensed fluid 100 to the hot end. The wick 32 is primarily needed when the fluid 100 must return by capillary action against gravity. The heat pipe 30 can sometimes be partially evacuated, a vent or other means known to those ordinarily skilled in the art, to adjust the boiling temperature of the working fluid 100.

[0031] In practice, heat pipes can achieve a thermal conductivity as high as one thousand times or more than that of a solid copper rod of the same dimensions. Heat pipes in production commonly have diameters of 3, 4, and 6 millimeters (metric), or 1/4 and 5/8 inches (English). Flat heat pipes have been produced as thin as 0.5 millimeters.

[0032] Heat pipes can be incorporated in several ways. For example, a center bolt 40 can be made hollow with an inner cavity (and incorporating a wick) so that it also acts as a heat pipe

(See FIG. 4). This concept synergistically takes advantage of the requirement for reduced bolt stiffness (and hence, reduced bolt cross-sectional area).

[0033] The overall bolt diameter can increase to optimize heat transfer while appropriately reducing the wall thickness to give the bolt 40 for a proper mechanical stiffness. Hollowing the bolt will mainly reduce stiffness as defined as the change in force divided by the change in length. This often needed because the bolt should not be stiffer than the ceramic. In this sense, the invention achieves a degree of synergy. Converting the center bolt 40 to a heat pipe can substantially increase effective thermal conductivity. This increase in thermal conductivity has the effect of significantly increasing the heat flow from the stack interior to the thermally conductive end masses, which then conduct heat to the surrounding fluid.

[0034] As shown in FIG. 5, a variant of the present invention involves drilling additional holes into ceramic rings 52 and inserting heat pipes 54. The heat pipes 54 can be comparatively small in diameter compared to the diameter of the stack. The cross-sectional area is proportional to the square of the diameter. Again, this use of the heat pipes 54 has the effect of substantially increasing heat flow from the ceramic interior, where the heat is most needed. The reduction in ceramic volume (and thus the reduction in available power and mechanical

stiffness) is minimal. This variant of the present invention has the advantage of avoiding redesign of the bolt 14.

[0035] Furthermore, increasing the length of the heat pipe 54 into the tail mass and the piston also increases the length of the "cool" region for the working fluid 100 to condense; thereby improving the performance of the heat pipe.

[0036] A further variant of the present invention (involving flex tensional transducers; **SEE FIG. 6**) involves a heat pipe either in the center or in multiple locations in the ceramic rings to conduct heat to the shell. This use of heat pipes in the flex tensional transducer has the advantage that both ends of the heat pipe are thermally "shorted" to the surrounding fluid, but the disadvantage of a shorter "cool" region for the heat pipe.

[0037] The foregoing description of the preferred embodiments of the invention has been presented for purposes of illustration and description only. It is not intended to be exhaustive nor to limit the invention to the precise form disclosed; and obviously many modifications and variations are possible in light of the above teaching. Such modifications and variations that may be apparent to a person skilled in the art are intended to be included within the scope of this invention as defined by the accompanying claims.

COOLING ACOUSTIC TRANSDUCER WITH HEATING PIPES

ABSTRACT

A transducer with a closed heat pipe is provided with a hot surface and a cold surface. The hot surface is in contact with the transducer interior and the cold surface is in contact with a cooler contact area. A fluid is used in the pipe which boils at the temperature of the hot surface and condenses at the temperature of the cold surface. A wick inside the heat pipe facilitates the return by capillary action of the condensed fluid to the hot end. The heat pipe can be evacuated to adjust the boiling temperature of the fluid. A variant involves drilling additional holes into ceramic rings and inserting heat pipes. Increasing the heat pipe length into the tail mass and the piston increases the cool region for the fluid to condense; thereby improving the performance of the transducer.

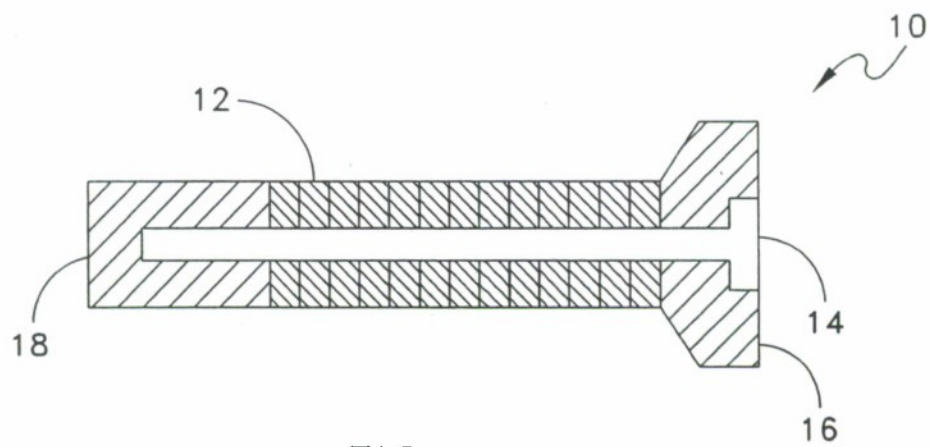


FIG. 1
(PRIOR ART)

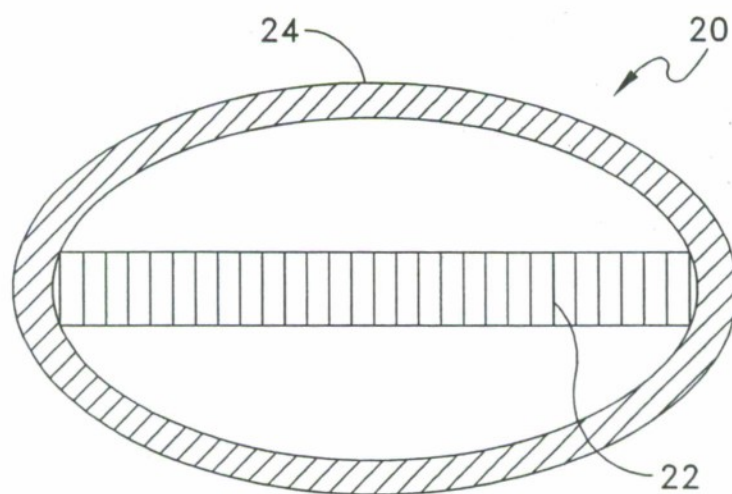
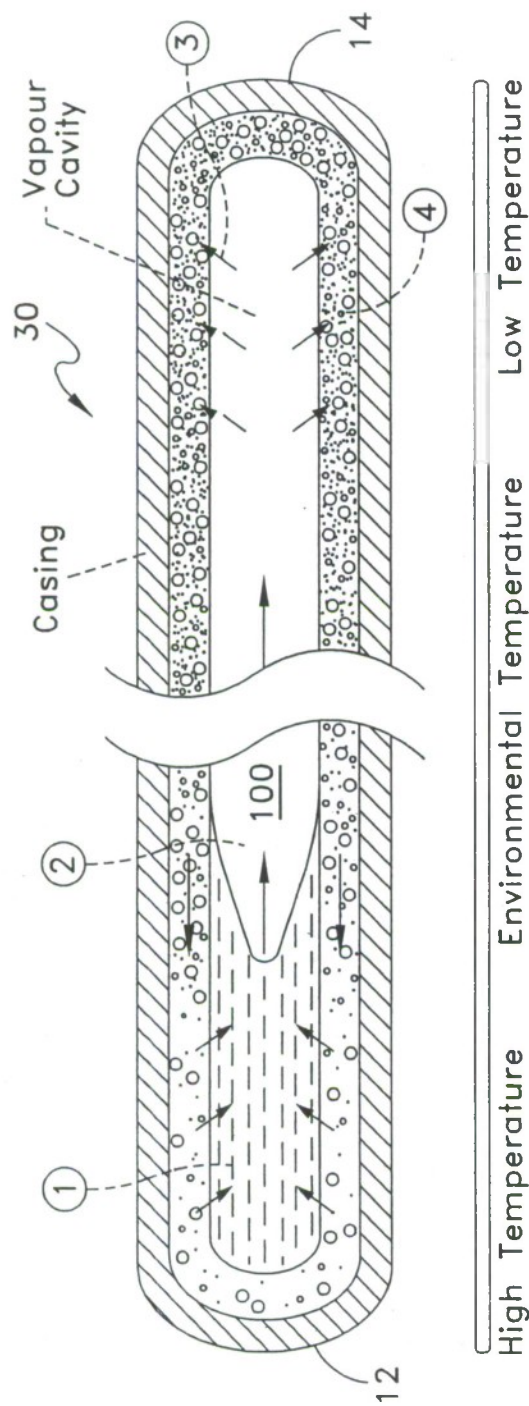


FIG. 2
(PRIOR ART)



HEAT PIPE THERMAL CYCLE

- ① Working fluid (100) evaporates to vapour absorbing thermal energy.
- ② Vapour migrates along cavity to lower temperature end.
- ③ Vapour condenses back to fluid and is absorbed by the wick (32), releasing thermal energy.
- ④ Working fluid flows back to higher temperature end.

FIG. 3

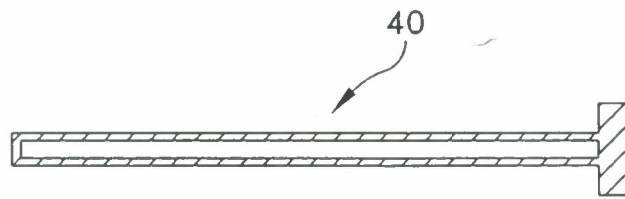


FIG. 4

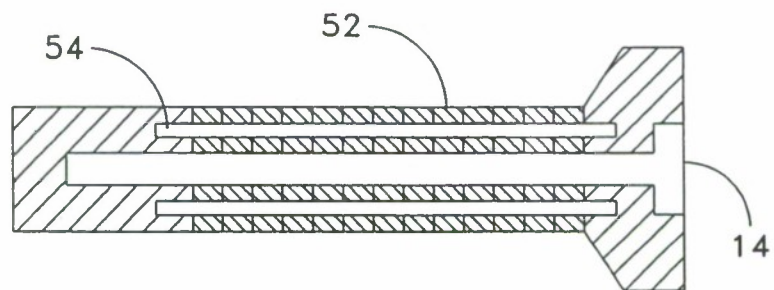


FIG. 5

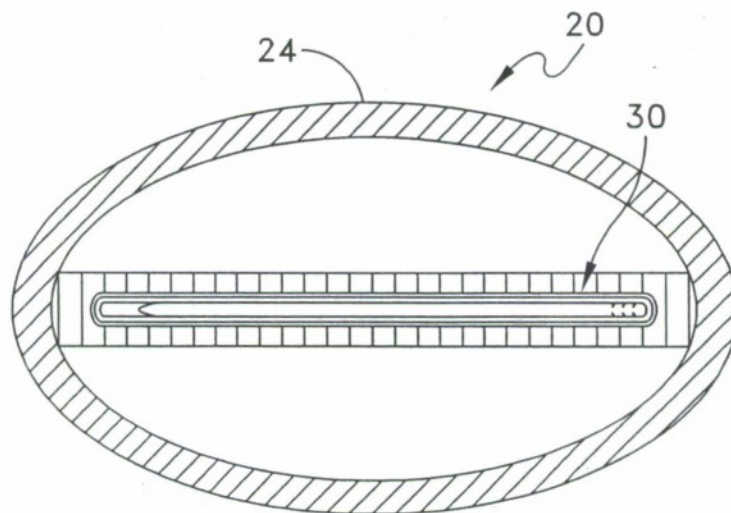


FIG. 6