



DEPARTMENT OF THE NAVY
NAVAL UNDERSEA WARFARE CENTER
DIVISION NEWPORT
OFFICE OF COUNSEL
PHONE: (401) 832-3653 FAX: (401) 832-4432
DSN: 432-3653



Attorney Docket No. 82377
Date: 8 August 2007

The below identified patent application is available for licensing. Requests for information should be addressed to:

PATENT COUNSEL
NAVAL UNDERSEA WARFARE CENTER
1176 HOWELL ST.
CODE 00OC, BLDG. 11
NEWPORT, RI 02841

Serial Number 11/229,424
Filing Date 12 September 2005
Inventor James R. Quartarone

If you have any questions please contact James M. Kasischke, Supervisory Patent Counsel, at 401-832-4230.

DISTRIBUTION STATEMENT
Approved for Public Release
Distribution is unlimited

20070824014

2

3

METHOD AND APPARATUS FOR IMPROVED ADHESION

4

AT AN INTERFACING SURFACE

5

6

STATEMENT OF GOVERNMENT INTEREST

7

8 The invention described herein may be manufactured and used
9 by or for the Government of the United States of America for
10 governmental purposes without the payment of any royalties
11 thereon or therefore.

11

12

BACKGROUND OF THE INVENTION

13

(1) Field of the Invention

14

15 The present invention relates to a method of improving and
16 increasing sustainable shear force capabilities at an interfacing
17 surface between a rigid material and a flexible material.

17

(2) Description of the Prior Art

18

19 It is well known to utilize a resilient bushing having a pair
20 of concentric rigid, typically metal sleeves. FIG. 1 is a prior
21 art example of a typical fastening arrangement 10 utilizing
22 adhesion at rigid-to-flexible interfacing surfaces. The inner
23 sleeve 13 is secured to one structural component 15 (for example
24 an input shaft) while the outer sleeve 18 is secured to another
structural component 20 (for example a propeller shaft). An

1 annular elastomeric insert 23 is concentrically positioned
2 between the rigid sleeves 13, 18. The interface between the
3 interfacing surfaces needs to transmit high shear and torque
4 loads. Previous designs would fail locally in shear due the high
5 torque applied during the functional application, and due to the
6 comparatively low shear which these devices could sustain.
7 Higher torque, shear, adhesion was needed at the rigid-to-
8 flexible material interfaces. The adhesion resistance to shear
9 failure at the surface interface 25 and 27 is what supports the
10 full torque during operation.

11 In the past, most interfacing surfaces 25 and 27 were
12 relatively smooth; that is, their surface roughness was generally
13 less than 170 RMS as measured by SAE Standard J448a. A method of
14 increasing adhesion resistance to shear failure was the addition
15 of a phosphate coating of various thicknesses. However, the
16 surface roughness did not exceed roughly 170 RMS.

17 It is also well known to increase shear force capabilities
18 between two surfaces by sandblasting the interfacing surfaces,
19 see for example U.S. Patent No. Re. 29,823 issued to Sievers et
20 al. (hereinafter referred to as Sievers et al.). Sievers et al.
21 disclose sandblasting at least one of the interfacing surfaces 25
22 and 27 and generating a generally even surface having a surface
23 roughness to between greater than 170 RMS and less than about 260
24 RMS.

1 The object of this invention is to improve and increase
2 sustainable shear force capabilities at an interfacing surface
3 between a rigid material (for example: a structural metal) and a
4 flexible material (for example: an elastomeric dampening
5 material).

6 Another object of the present invention is to improve and
7 increase the sustainable shear force capabilities at an
8 interfacing surface without internal or external fastening
9 devices such as bolts, screws, washers, and the like.

10 Yet another object of the present invention is to improve
11 and increase the sustainable shear force capabilities at an
12 interfacing surface in a smaller, more compact volume than
13 previous methods.

14 Yet another object of the present invention is to improve
15 and increase the sustainable shear force capabilities at an
16 interfacing surface with a simplistic (that is without a large
17 number of parts) and economy.

18

19

SUMMARY OF THE INVENTION

20 A method and apparatus for improving resistance to adhesion
21 failure between two interfacing surfaces utilizing by producing a
22 particular shape and roughness is disclosed. The present
23 invention utilizes both coarse and fine surface roughness
24 producing techniques. The surface, to be treated, has a coarse

1 (large scale) surface roughness component on the order of about
2 three to four orders of magnitude greater than the fine (small
3 scale) surface roughness component. The invention uses a
4 plurality of superimposed roughnesses rather than one range of
5 roughness, as seen in prior art. The appropriate surfaces of the
6 rigid part are roughened (e.g., by machining, casting, molding,
7 soundblasting) to yield surfaces having both small (fine) and
8 large (coarse) scale roughness components. The large-scale
9 roughness component is about three orders of magnitude greater
10 than the small scale roughness component.

11

12

BRIEF DESCRIPTION OF THE DRAWINGS

13 These and other features and advantages of the present
14 invention will be better understood in view of the following
15 description of the invention taken together with the drawings
16 wherein:

17

FIG. 1 is a cross-section of a prior art bushing;

18

FIG. 2 is a cross-section of one embodiment of the present
19 invention shown as a bushing, and FIG. 2A is an enlarged detail
20 of a zone of an interface between a rigid member and an
21 elastomeric member circumscribed by phantom lines 2A, FIG. 2;

22

FIG. 3 is an exploded perspective view of a portion on

23

FIG. 2;

1 FIGS. 4A and 4B, respectively, are elevation views of the
2 relative scales of the fine and coarse roughnesses shown
3 separately;

4 FIG. 5 is an elevation view of one embodiment of the present
5 invention showing the coarse and fine surface roughness
6 components combined;

7 FIG. 6 is a front view of an engine incorporating engine
8 mounts utilizing the present invention;

9 FIG. 7 is a plan view of the present invention used with a
10 marine/aeronautical application;

11 FIGS. 8A-8I are elevation views showing different surface
12 configurations of the present invention; and

13 FIG. 9A is partially a side view and partially a sectional
14 view of an elastomeric tread wheel, and FIG. 9B is an end view of
15 this wheel.

16

17 **DESCRIPTION OF THE PREFERRED EMBODIMENT**

18 Referring to FIGS. 2 and 3, one embodiment of a bushing 10'
19 according to the present invention is shown utilizing a generally
20 an elastomeric material 30, an outer rigid member 33, and an
21 inner rigid member 35. The elastomeric material 30 can be any
22 well-known elastomeric material and would be selected based upon
23 the desired dampening and isolation properties, all of which is
24 well within the knowledge of those skilled in the art. The

1 elastomeric material is moldingly bonded to the rigid member 33,
2 35 at confronting interfacing surfaces 39, 40.

3 In a preferred embodiment, a 50 durometer fluorosilicone
4 elastomer was used. However, 40, 50, 60, and 70 durometer
5 elastomers have also been tested. It should be stressed, however,
6 that the choice of the elastomeric material 30 will depend on the
7 circumstances of its intended use and is not limited in any way
8 to these stated durometer values or to fluorosilicone elastomers.

9 The inner and outer rigid members 33 and 35 may also be
10 selected from any material. The material chosen will again
11 depend upon the circumstances in which it is intended to be used.

12 Typically, the inner and outer rigid members 33 and 35 are made
13 from metal (e.g., aluminum); however, the material can be any
14 material having the necessary rigidity such as steel, plastic,
15 laminates, composites, ceramics, biological material (bones,
16 teeth, tusks), etc. Optionally, the inner and outer rigid
17 members 35 and 33 may be anodized. The decision to anodize the
18 inner and outer rigid members 35 and 33 will depend on a
19 combination of the material chosen and the operating conditions.

20 The inner rigid member 35 has an outer surface 40 while the
21 outer rigid member 33 has an inner surface 39. The outer surface
22 40 and the inner surface 39 are machined, or otherwise created,
23 to have a coarse surface roughness component. In one preferred
24 embodiment, the coarse surface roughness component was created

1 first, using a standard industrial milling machine, and the fine
2 roughness component was produced afterwards, by sandblasting.

3 The outer surface 40 and/or inner surface 39 are sandblasted or
4 otherwise created using standard industrial techniques.

5 Most typically, the dual coarse-fine roughness is produced
6 on the rigid material surface by two separate operations; e.g., a
7 machining operation followed by a blasting operation. However,
8 alternatively, both coarse and fine roughness components may be
9 produced at the same time; e.g., by a casting operation for
10 metals, or by a molding operation for rigid plastics. This
11 invention uses a plurality of roughness components working
12 cooperatively. In one embodiment, the coarse roughness component
13 was created using a milling operation. In another embodiment,
14 the coarse roughness component was created using an elox
15 operation. In one embodiment, the fine roughness component was
16 created by grit blasting with aluminum oxide.

17 The coarse roughness component would generally be within the
18 range of 50,000 to 1,000,000 micro-inches. Coarse roughness
19 components may center around any coarse roughness (e.g., 50,000,
20 200,000, etc.) within this 50,000 to 1,000,000 micro-inch range.

21 The fine roughness component would generally be within the range
22 of 100 to 2,000 micro-inches. Fine roughness components may
23 center around any fine roughness (e.g., 200, 550, 1,000, etc.)
24 within this 100 to 2,000 micro-inch range. In one particular

1 successful embodiment, a fine roughness component of 550 micro-
2 inches was specified with a tolerance of ± 150 micro-inches
3 producing an actual measured fine roughness component between 400
4 to 700 micro-inches.

5 The coarse roughness component is three orders of magnitude
6 greater than the fine roughness component; i.e., the ratio of
7 coarse roughness component to fine roughness component will, in
8 one embodiment, be 1,000. Examples of various embodiments of
9 this invention, which demonstrate the relationship between the
10 coarse roughness component and the fine roughness component, are
11 presented.

12 The outer surface 40 and/or inner surface 39 are
13 characterized by having surface roughness components that include
14 both coarse 50, (large scale), FIG. 4B, and fine 51 (small
15 scale), FIG. 4A roughness surfaces relative to each other. It
16 should be noted that any surface which needs to improve and
17 increase sustainable adhesives force capabilities at an
18 interfacing surface where a flexible material confronts a rigid
19 material could utilize the present invention.

20 Fig. 5 shows one embodiment 55 of this combination of coarse
21 50 and fine 51 surface roughness. While the entire coarse
22 roughness surface 50 is shown in combination with the fine
23 roughness surface 51, the coarse roughness surface 50 may
24 optionally contain only smaller or partial regions having the

1 fine roughness surface 51, and also regions of the fine roughness
2 surface 51 may optionally contain no coarse roughness component
3 (e.g., near end regions of the interfacing surfaces).

4 The coarse surface roughness component 50 is of the order of
5 three orders of magnitude greater than the roughness of the fine
6 surface roughness component 51. In a preferred embodiment,
7 coarse surface 50 has a surface roughness component of about
8 200,000 micro inches and a fine surface 51 roughness component
9 has a surface roughness of about 500 micro inches.

10 In another embodiment, a coarse surface 50 roughness of
11 300,000 micro-inches and a fine surface 51 roughness of 200
12 micro-inches was used. The interface 55 having both coarse 50
13 and fine 51 surface roughness components allows the interface to
14 sustain greater shear and other forces than surfaces having only
15 fine or coarse roughness surfaces.

16 The coarse and fine surface 55 can be applied to any device
17 with a rigid-to-flexible interface that needs to resist shear,
18 tension, torsion, compression, or any disturbing steady-state or
19 variable force or forces. The coarse and fine surface 55 could
20 be used on virtually any type of anti-shock, vibration, or noise
21 mount; e.g., automobile engine mounts (FIG. 6) or
22 marine/aeronautical propeller shaft applications (FIG. 7). FIG.
23 6 mounted engine assembly 60 shows engine 61 mounted on engine
24 mounts 67, supported on frame structure 68. Rigid material 62

1 and 63 have coarse-fine surfaces 55 applied to interface 64 and
2 65. Flexible material 66 has been sandwiched and moldingly
3 bonded between rigid materials 62 and 63 (i.e., bonded at
4 interfaces 64 and 65).

5 FIGS. 9A and 9B depict an elastomeric treaded wheel 90
6 showing elastomeric material 96 moldingly bonded to a circular
7 rigid material wheel rim 92, at confronting interfacing surface
8 94.

9 FIGS. 8A-8I show some of the different possible shapes of
10 surface roughness 81-89. For purposes of the present invention,
11 the surface roughness can be any suitable shape. Graphically
12 portraided shapes 81, 83, 85, and 87, FIGS. 8A, 8C, 8E and 8G
13 depict simple sine-wave or approximate sine-wave shapes as the
14 shape of the coarse surface 50 in addition to the saw-tooth shape
15 82 as shown in FIGS. 4A-B and 5. FIG. 8B shows a mix of coarse
16 50 and fine 51 surface roughness. The difference in magnitude
17 between the coarse 50 and fine 51 surface roughness is so large
18 that the fine 51 surface roughness cannot be readily visually
19 detected in FIG. 8B. However, the other views, FIGS. 5, 8D, 8F,
20 and 8H, show both coarse and fine components.

21 FIG. 8D shows one embodiment 84 wherein a rough saw-tooth
22 fine roughness 51 is superimposed on a coarse sine-wave surface
23 roughness 50. Shape 86, FIG. 8F shows a slightly larger and more
24 random fine surface roughness 51 saw-tooth profile superimposed

1 on a coarse sine-wave roughness 50. Shape 88, FIG. 8H shows yet
2 another embodiment wherein a slightly random fine roughness 51
3 saw-tooth profile is superimposed on a fairly regular coarse
4 roughness 50 saw-tooth shape. Lastly, shape 89, FIG. 8I shows an
5 embodiment utilizing at least three combinations of surface
6 roughness. This combination is referred to as a fine-mid-coarse
7 blend. Generally, the mid-surface roughness component would be
8 20-200 times greater than the fine surface roughness component,
9 and the coarse surface roughness component would be 20-200 times
10 greater than the mid-surface roughness component. In one
11 embodiment, the fine surface roughness component is approximately
12 200 RMS micro-inches, the "mid" surface roughness component is
13 approximately 4,000 RMS micro-inches, and the coarse surface
14 roughness component is approximately 80,000 RMS micro-inches. In
15 another embodiment of a fine-mid-coarse roughness combination,
16 the "fine" surface roughness is approximately 150 RMS micro-
17 inches, the "mid" surface roughness is approximately 7,500 RMS
18 micro-inches, and the "coarse" surface roughness is approximately
19 375,000 RMS micro-inches.

20 Rigid components 62, 72, would be designed using well-known
21 engineering criteria for design and performance and would be
22 manufactured using any suitable standard art. Flexible material
23 30, 66, 76, would be selected using well-known engineering
24 criteria for required characteristics (e.g., damping, strength,

1 durometer value, shock, and vibration properties) and would be
2 manufactured using any suitable molding process which leaves or
3 encourages the desired resilient properties and facilitates
4 bonding to interfacing surfaces 64. It would be advantageous, as
5 with any molding process, to pay attention to eliminating voids
6 (e.g., gas, air bubbles) and contaminants (e.g., dirt, grease) at
7 the interface 64. As with standard methods, proper surface
8 preparation is advised. It should be noted that an interface
9 with this improved method of bonding geometry will be stronger
10 than other methods at all levels of cleanliness.

11 In light of the above, it is therefore understood that
12 within the scope of the appended claims, the invention may be
13 practiced otherwise than as specifically described.

2

3

METHOD AND APPARATUS FOR IMPROVED ADHESION

4

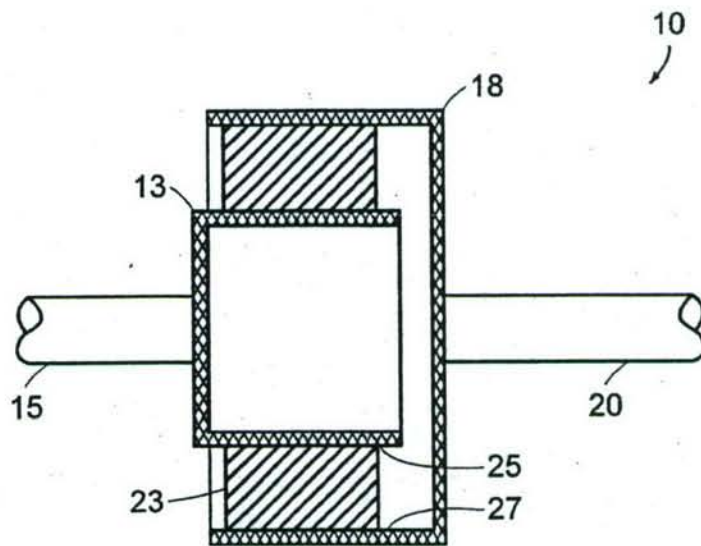
AT AN INTERFACING SURFACE

5

6

ABSTRACT OF DISCLOSURE

7 An apparatus for improving and increasing sustainable shear
8 force capabilities at an interfacing surface between a rigid
9 material and a flexible material is disclosed. The interface
10 includes at least two superimposed components having different
11 surface roughnesses. A coarse surface roughness component has a
12 surface roughness about 3 orders of magnitude greater than that
13 of a fine surface roughness component. The interface is useful
14 for all types of devices with a rigid-to-flexible interface that
15 needs to resist shear, tension, torsion, compression, or any
16 disturbing steady-state or variable force or forces.



PRIOR ART
FIG. 1

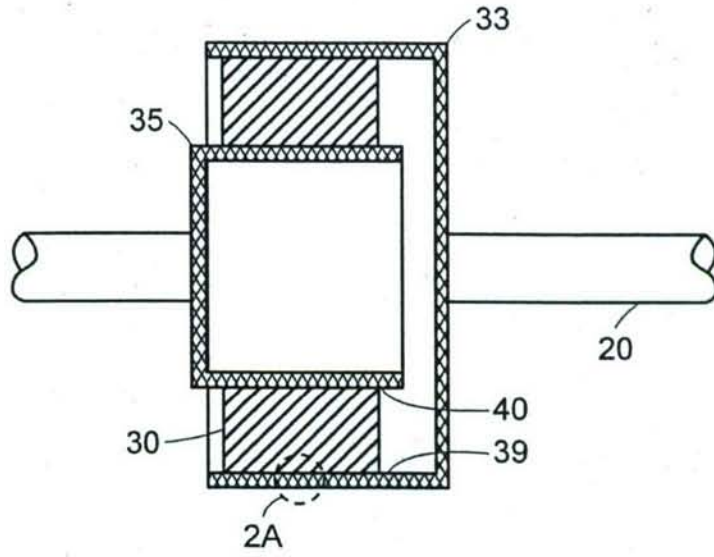


FIG. 2

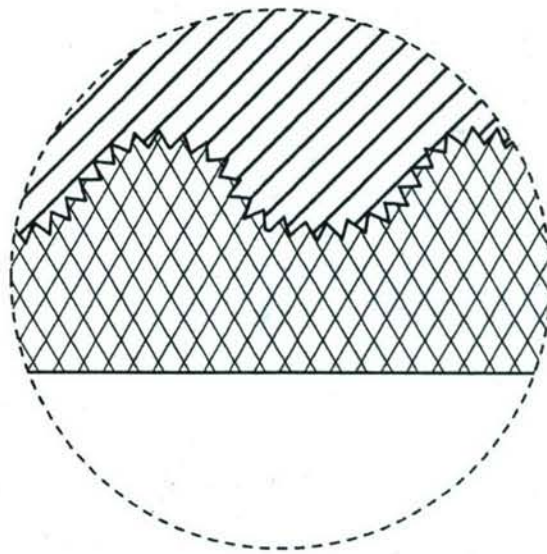


FIG. 2A

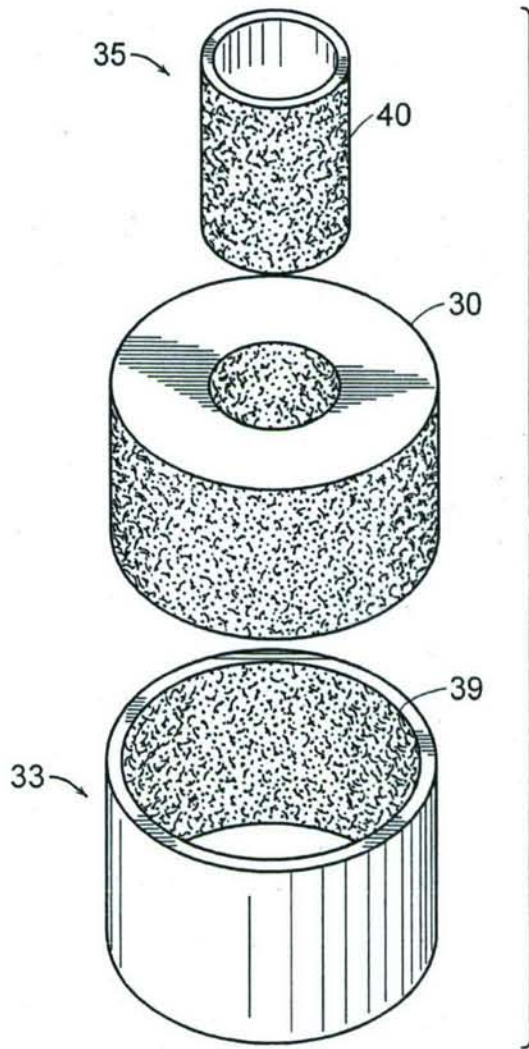


FIG. 3



FIG. 4A

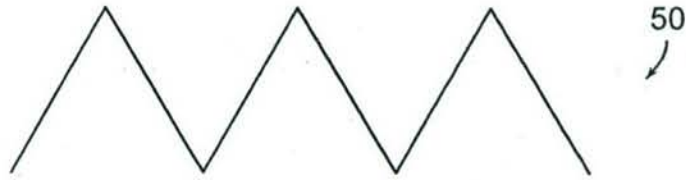


FIG. 4B

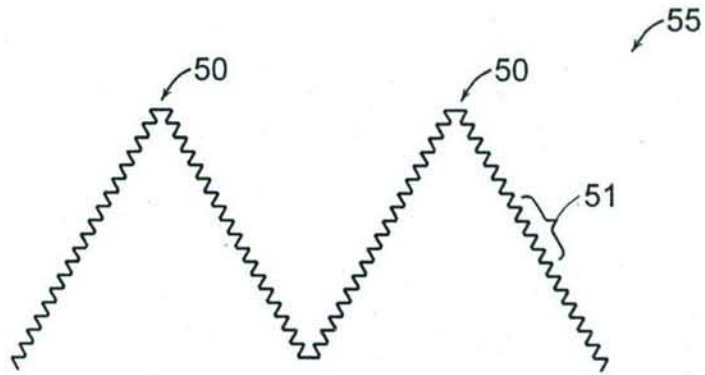


FIG. 5

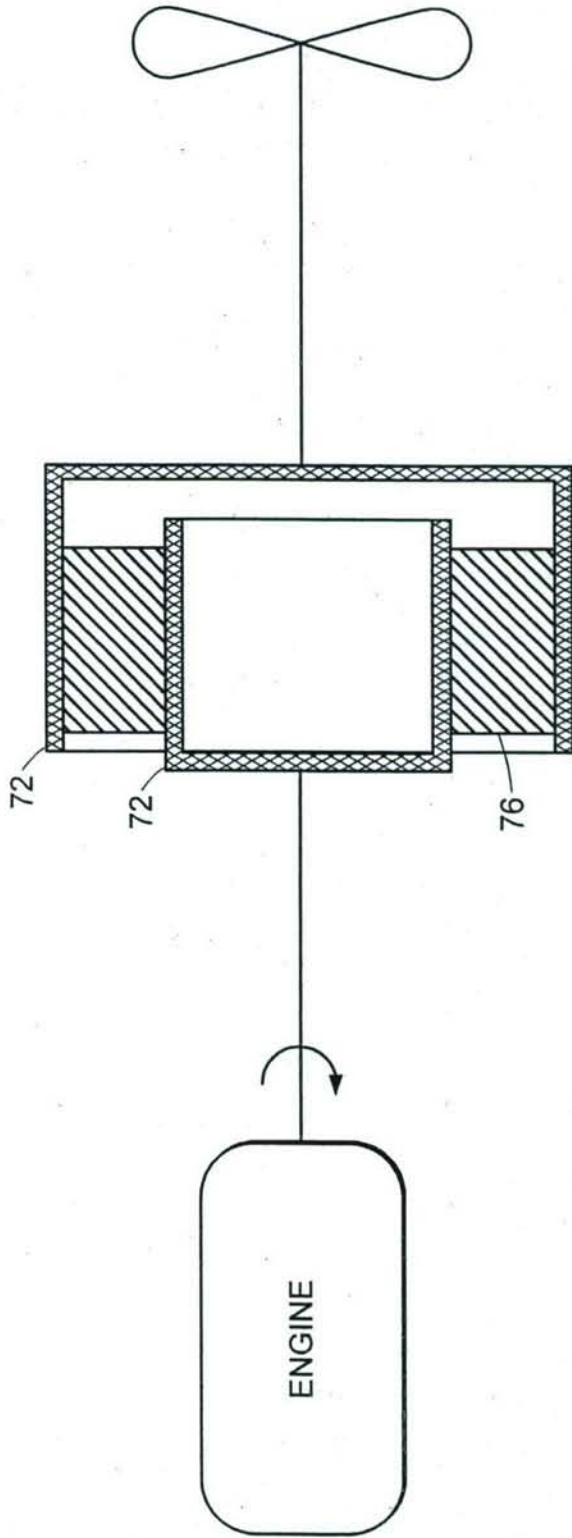


FIG. 7

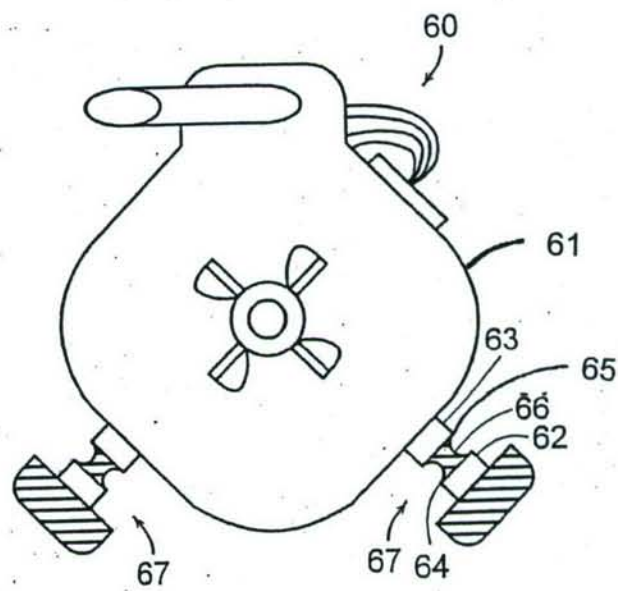


FIG. 6



FIG. 8A



FIG. 8B



FIG. 8C



FIG. 8D



FIG. 8E



FIG. 8F



FIG. 8G



FIG. 8H



FIG. 8I

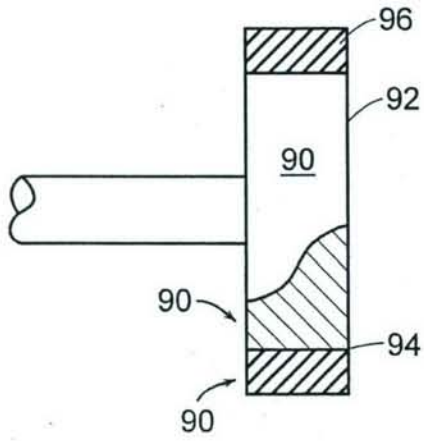


FIG. 9A

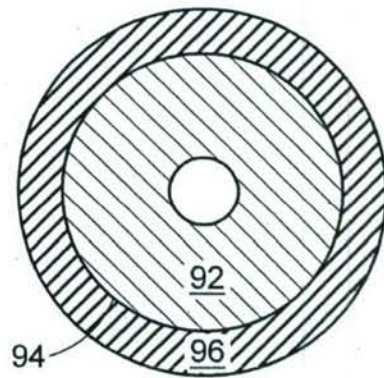


FIG. 9B