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FLOW REVERSAL SYSTEM FOR AXIAL FAN

TO ALL WHOM IT MAY CONCERN

BE IT KNOWN THAT GUY F. BORGES, citizen of the United States of America, employee of the United States Government and resident of Somerset, County of Bristol, Commonwealth of Massachusetts, has invented certain new and useful improvements entitled as set forth above of which the following is a specification:

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FLOW REVERSAL SYSTEM FOR AXIAL FAN

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STATEMENT OF GOVERNMENT INTEREST

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The invention described herein may be manufactured and used by or for the Government of the United States of America for Governmental purposes without the payment of any royalties thereon or therefore.

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CROSS REFERENCE TO OTHER RELATED APPLICATIONS

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Not applicable.

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BACKGROUND OF THE INVENTION

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(1) Field of the Invention

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The present invention relates generally to axial fans, and more particularly to a flow reversal system for an axial fan that quickly reverses the flow of fluid through the fan without changing the rotational direction or speed of the fan.

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(2) Description of the Prior Art

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Fan arrangements in conventional air handling systems utilize motor driven fan wheels that draw ventilation air from the outdoors through ducts, louvers and dampers. Although these fan systems may be able to vary their speed and capacity, they almost always have significant amounts of rotational inertia in the fan wheel/motor assembly that can require minutes for the fan wheel to completely stop rotating after the motor has been turned

1 off. However, in situations where air contaminants are detected,  
2 it is desirable to shut down the air flow as soon as possible and  
3 even reverse same to purge the contaminated air from an indoor  
4 environment.

5 Typically, when contaminants are detected, fan motors are  
6 shut off and dampers are used to shut off an air flow in a duct.

7 The use of dampers to rapidly seal off air flow is problematic  
8 since most fast-acting damper assemblies are not capable of a  
9 completely tight seal. Further, damper seals experience very  
10 high pressure excursions when a fan discharge is rapidly closed  
11 off. These factors make it extremely difficult to devise a  
12 scheme that quickly stops the flow of outdoor air being drawn  
13 into an air handling system upon detection of contaminants in  
14 that air stream.

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#### SUMMARY OF THE INVENTION

17 Accordingly, it is an object of the present invention to  
18 provide a system that can quickly reverse the flow of fluid  
19 caused by a fan.

20 Another object of the present invention is to provide a flow  
21 reversal system for an axial fan that can change the flow of  
22 fluid through the fan without changing the rotational speed or  
23 direction of the fan.

24 Other objects and advantages of the present invention will  
25 become more obvious hereinafter in the specification and  
26 drawings.

1           In accordance with the present invention, a flow reversal  
2 system is provided for use with an axial fan having a drive shaft  
3 that rotates about a longitudinal axis thereof. A blade support  
4 wheel is coupled at a central portion thereof to the drive shaft  
5 for rotation therewith. A disk is positioned about the drive  
6 shaft adjacent the blade support wheel. A torsion spring is  
7 coupled to the blade support wheel and to the disk such that,  
8 when the torsion spring is in tension, the disk is poised for a  
9 relative rotation about the drive shaft. The relative rotation  
10 is relative to the blade support wheel. A lock couples the disk  
11 to the blade support wheel when the torsion spring is in tension  
12 so that rotation of the blade support wheel causes corresponding  
13 rotation of the disk. Each of a plurality of fan blades has a  
14 stem rotationally supported by the blade support wheel so that a  
15 rotational position of the stem sets blade pitch for a  
16 corresponding one of the fan blades. Means are provided to  
17 couple each stem to the disk such that rotation of the drive  
18 shaft causes rotation of the fan blades about the drive shaft.  
19 The coupling of the stem to the disk also positions the fan  
20 blades to generate a flow therethrough when the drive shaft  
21 rotates with the lock coupling the disk to the blade support  
22 wheel while the torsion spring is in tension. Release means are  
23 coupled to the lock means when it is desired to uncouple the lock  
24 from the disk and blade support wheel. Specifically, when the  
25 release means is activated, the disk experiences relative  
26 rotation about the drive shaft as tension in the torsion spring  
27 is released. The relative rotation of the disk causes rotation

1 of each stem thereby re-positioning the fan blades to reverse the  
2 flow therethrough as the drive shaft rotates.

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#### BRIEF DESCRIPTION OF THE DRAWINGS

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10 Other objects, features and advantages of the present  
11 invention will become apparent upon reference to the following  
12 description of the preferred embodiments and to the drawings,  
13 wherein corresponding reference characters indicate corresponding  
14 parts throughout the several views of the drawings and wherein:

15 FIG. 1 is a schematic view of a flow reversal system for an  
16 axial fan in accordance with an embodiment of the present  
17 invention; and

18 FIG. 2 is an isolated head-on view of a single fan blade  
19 illustrating the relationship between the blade support wheel and  
20 geared disk as viewed along line 2-2 in FIG. 1.

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#### DESCRIPTION OF THE PREFERRED EMBODIMENT(S)

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Referring now to the drawings, and more particularly to FIG.

1, the flow reversal system for an axial fan in accordance with

the present invention is depicted schematically. As is known in

the art, an axial fan is one that has a drive shaft 100 that

rotates about its longitudinal axis 102 as shown by drive arrow

104. Powered rotation of drive shaft 100 can be achieved by a

motor (not shown) coupled directly to drive shaft 100 or

indirectly to drive shaft 100 by means of a belt or chain.

Accordingly, it is to be understood that the type of axial fan

and drive source are not limitations of the present invention.

1 A wheel 10 is coupled at coupling 12 to drive shaft 100 for  
2 rotation therewith. Wheel 10 is rigid and has an annular flange  
3 14 at its peripheral edge. Rotatably supported in annular flange  
4 14 are a plurality of spaced apart fan blade stems 20. More  
5 specifically, as best seen in FIG. 2, each stem 20 passes through  
6 annular flange 14 and is supported in a bushing 22 that supports  
7 rotation of stem 20 as indicated by arrow 24. Coupled to each  
8 stem 20 outside of annular flange 14 is a fan blade 26, the size  
9 and shape of which are not limitations of the present invention.

10 Although only two blade/stem combinations are shown, it is to be  
11 understood that additional blades/stems can be used without  
12 departing from the scope of the present invention. Coupled to  
13 each stem 20 inside of annular flange 14 is a bevel gear 28.

14 A disk 30 is disposed about drive shaft 100 but is not  
15 directly coupled thereto. That is, the rotation of drive shaft  
16 100 is not directly coupled to disk 30. One face of disk 30 is  
17 configured with gear teeth 32 (as best seen in FIG. 2) that  
18 engage the teeth of bevel gear 28. As a result, independent  
19 rotation of disk 30 about drive shaft 100 results in rotation  
20 (indicated by arrow 24) of each bevel gear 28/stem 20/fan blade  
21 26 combination. Conversely, the prevention of any independent  
22 rotation of disk 30 about drive shaft 100 fixes the position of  
23 each bevel gear 28/stem 20/fan blade 26 combination thereby  
24 fixing the pitch of each fan blade 26.

25 A torsion spring ("TS" in FIG. 1) 34 is coupled to each of  
26 wheel 10 and disk 30. Torsion spring 34 is any spring device  
27 that, when in tension, can generate a rotational force on disk 30

1 such that disk 30 can rotate about drive shaft 100 relative to  
2 wheel 10. A variety of such spring devices are known in the art  
3 and include, but are not limited to, spiral or helical clock-type  
4 springs disposed about drive shaft 100, coil spring(s)  
5 cooperating between fixed mounts on each of wheel 10 and disk 30,  
6 etc.

7 According to the present invention, the position of disk 30  
8 relative to wheel 10 is limited to one of two (pitch) positions.

9 In this way, the position of each fan blade 26 is limited to one  
10 of two positions. The first of these positions is defined when  
11 torsion spring 34 is placed in tension, i.e., disk 30 is rotated  
12 about drive shaft 100 until fan blades 26 are positioned such  
13 that their rotation about drive shaft 100 produces a flow of the  
14 surrounding fluid medium (e.g., air or other gas) in a first  
15 direction. By way of illustrative example, the first flow  
16 direction is indicated by directional arrow 200.

17 It is necessary to fix or lock disk 30 in position relative  
18 to wheel 10 when torsion spring 34 is in tension in order to  
19 maintain the first position of fan blades 26. One way of doing  
20 this is to provide a plate 40 that is disposed about drive shaft  
21 100 such that plate 40 and drive shaft 100 are not coupled to one  
22 another. Extending from plate 40 and parallel to longitudinal  
23 axis 102 are a plurality (e.g., two are shown in FIG. 1) of  
24 locking pins 42 that are long enough to engage receiving holes  
25 (not shown) formed in the faces of each of disk 30 and wheel 10.  
26 By locking wheel 10 and disk 30 together with torsion spring 34  
27 in tension, rotation of drive shaft 100 is translated through



1 disk 30 to each of fan blades 26 to generate fan flow 200. Note  
2 that plate 40/pins 42 rotate about drive shaft 100 when wheel 10  
3 and disk 30 are locked together.

4 The present invention provides for the complete reversal of  
5 fan flow 200 without requiring any change in the rotational speed  
6 or direction of drive shaft 100. By way of illustrative example,  
7 one or more actuators 50 (e.g., electromagnetic, hydraulic, etc.)  
8 having actuator rods 52 extending therefrom can be configured  
9 with end plates 52A positioned such that plate 40 is not engaged  
10 by end plates 52A while fan flow 200 is being generated.  
11 However, when fan flow 200 must be reversed (i.e., such that  
12 reversed fan flow 202 is generated) actuators 50 are activated  
13 so that rods 52 are axially retracted whereby end plates 52A  
14 engage plate 40. Engagement of plate 40 can be realized by this  
15 or other types of mechanical engagement of the peripheral edge or  
16 other portions of plate 40. However, it is to be understood that  
17 such engagement need not be mechanical. For example,  
18 electromagnetic forces could be applied to plate 40 in order to  
19 move it axially along drive shaft 100.

20 Regardless of the particular choice of motive force, flow  
21 reversal is achieved when plate 40 is pulled in a direction  
22 parallel to longitudinal axis 102 so that pins 42 are disengaged  
23 from wheel 10 and disk 30. Once pins 42 are disengaged, disk 30  
24 rotates about drive shaft 100 (relative to wheel 10) under the  
25 tension of torsion spring 34. The tension supplied by torsion  
26 spring 34 should be sufficient to rotate disk 30 to a position  
27 that correspondingly rotates each fan blade 26 to a position that

1 generates reversed fan flow 202 as drive shaft 100 continues to  
2 rotate.

3       The advantages of the present invention are numerous. Fan  
4 flow is quickly reversed since no motor deceleration or change of  
5 direction is required. Thus, the present invention is ideally  
6 suited for use in ventilation systems where change in conditions  
7 may warrant a reversal of fan flow. Accordingly, a variety of  
8 condition sensors may be placed in and/or remotely with respect  
9 to the fan flow to detect such conditions. Detection of such  
10 condition(s) can then be used to trigger activation of actuators  
11 50. The condition sensor(s) can include sensor(s) 60 placed in  
12 fan flow 200 and/or sensor(s) 62 placed remotely with respect to  
13 fan flow 200. In terms of ventilation systems, sensors 60 and/or  
14 62 can be contaminant sensors that trigger activation of  
15 actuators 50 when contaminants are detected. Note that the  
16 output of sensors 60 and 62 could also be used to trigger other  
17 components that are affected by fan flow reversal. For example,  
18 filters (not shown) might normally positioned in the low-pressure  
19 side of fan blades 26 during fan flow 200. However, the presence  
20 of such filters during reversed fan flow 202 may impede such  
21 flow. Accordingly, the output of sensors 60 and 62 could also be  
22 used to trigger components/systems used to move or remove such  
23 filters or other obstructions during reversed fan flow 202.

24       It will be understood that many additional changes in the  
25 details, materials, steps and arrangement of parts, which have  
26 been herein described and illustrated in order to explain the  
27 nature of the invention, may be made by those skilled in the art

- 1 within the principle and scope of the invention as expressed in
- 2 the appended claims.

1 Attorney Docket No. 84169

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FLOW REVERSAL SYSTEM FOR AXIAL FAN

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ABSTRACT OF THE DISCLOSURE

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An axial fan has a blade support wheel coupled to a drive shaft. A disk is positioned about the drive shaft adjacent the wheel. A torsion spring is coupled to the wheel and to the disk.

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When the torsion spring is in tension, the disk is poised for a relative rotation about the drive shaft. A lock couples the disk

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to the wheel when the torsion spring is in tension. Fan blades

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are supported by the wheel, and are coupled to the disk.

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Rotation of the drive shaft rotates the fan blades generating a

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specific flow direction. To reverse the flow direction, a

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release mechanism uncouples the lock from the disk and wheel

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causing the disk to experience rotation about the drive shaft as

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tension in the torsion spring is released. The disk's relative

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rotation re-positions the fan blades reversing the flow direction

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as the drive shaft continues to rotate.

FIG. 1

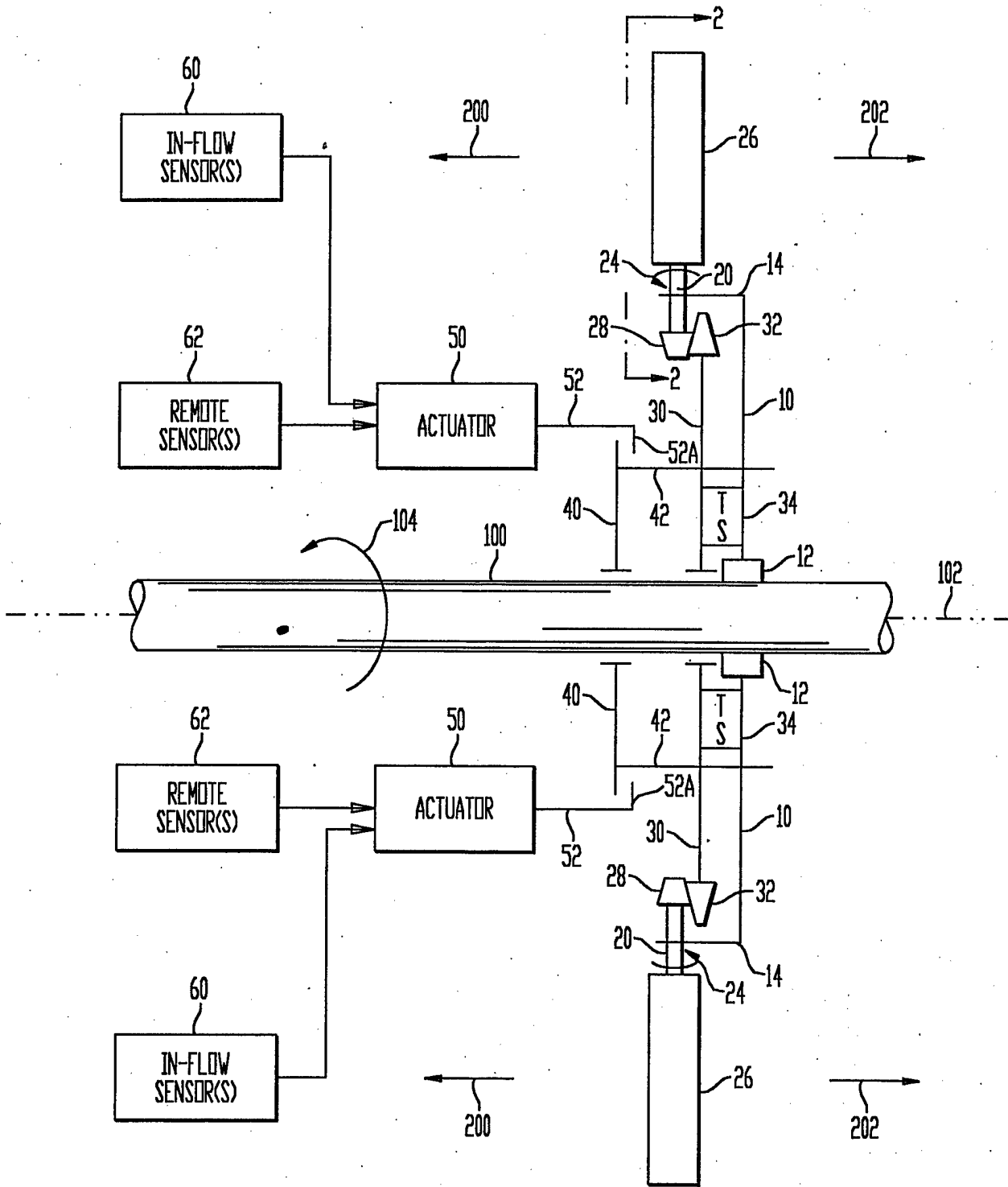


FIG. 2

