

DEPARTMENT OF THE NAVY

OFFICE OF COUNSEL NAVAL UNDERSEA WARFARE CENTER DIVISION 1176 HOWELL STREET NEWPORT RI 02841-1708

IN REPLY REFER TO:

Attorney Docket No. 83086 Date: 30 October 2003

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PATENT COUNSEL NAVAL UNDERSEA WARFARE CENTER 1176 HOWELL ST. CODE 00OC, BLDG. 112T NEWPORT, RI 02841

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DISTRIBUTION STATEMENT A

Approved for Public Release Distribution Unlimited

ASYMETRICALLY CONTOURED ELASTOMERIC DISK

TO ALL WHOM IT MAY CONCERN:

BE IT KNOWN THAT (1) WILLIAM P. BARKER and (2) JEFFREY R. MILBURN, employees of the United States Government, citizens of the United States of America, and residents of (1) Bristol, County of Bristol, State of Rhode Island (2) Providence, County of Providence, State of Rhode Island have invented certain new and useful improvements entitled as set forth above of which the following is a specification:

MICHAEL P. STANLEY, ESQ. Reg. No. 47108 Naval Undersea Warfare Center Division, Newport Newport, RI 02841-1708 TEL: 401-832-4736 FAX: 401-832-1231

> DISTRIBUTION STATEMENT "A" Approved for Public Release; distribution is unlimited.



1	Attorney Docket No. 83086
2	
3	ASYMETRICALLY CONTOURED ELASTOMERIC DISK
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5	STATEMENT OF GOVERNMENT INTEREST
6	The invention described herein may be manufactured and used
7	by or for the Government of the United States of America for
8	governmental purposes without the payment of any royalties
9	thereon or therefor.
10	
11	BACKGROUND OF THE INVENTION
12	(1) Field of the Invention
13	The present invention relates to a device for use in an
14	elastomeric vehicle launching system, and more particularly to
15	an elastomeric disk for the storage of elastic energy
16	convertible to impulse fluid energy with the impulse fluid
17	capable of ejecting or launching vehicles from the system into a
18	liquid medium.
19	(2) Description of the Prior Art
20	Impulse fluid flows are used to launch vehicles from
21	submarine platforms. Elastomeric ejection systems have been
22	developed which store impulse fluid in a charged elastomeric
23	bladder or against the pressure surface of a distended
24	elastomeric disk. In the operation of an ejection system with

an elastomeric disk, a recharge pump of the system draws water
from the ocean such that the inflow of water has a pressure that
distends or expands the disk. The elastic deformation of the
disk by expansion results in a storage of energy by the disk.
Once a predetermined amount of energy is stored, the recharge
pump is shut off.

7 In order to launch a vehicle, a slide valve for the 8 designated torpedo tube is opened. The opening action of the 9 slide valve allows instantaneous porting water from the expanded 10 disk to the torpedo tube with the porting water pressure capable 11 of launching weapons from the tube.

Typical disks used for elastomeric ejection systems have a 12 flat, ellipsoidal, spherical, or other symmetrically contoured 13 shape. The purpose of a symmetrical contoured shape is to 14 15 distribute strain energy across the disk during expansion. For example, the prior art disk 2 shown in FIG.1 and shown in the 16 cross-sectional view of FIG. 2 has a symmetrical contour about a 17 central plane 4. When the prior art disk 2 expands, as shown in 18 FIG. 3, the expansion of the disk 2 outward is the greatest at 19 the center axis 6 of the disk 2. The disk 2 must be clamped or 20 attached at its edges 8, 10 to the supporting structure 12 in 21 order to expand outward. 22

The problem with the clamping of the disk 2 to the supporting structure 12 is that the expansion of the disk

consequently puts a significant material strain at the periphery 1 of the disk 2. While there is a contact strain with the 2 supporting structure 12, the material strain is greater on the 3 pressure surface 14 of the disk 2 specifically at the point 16 4 where the disk 2 bends toward deformation of the disk. This 5 material strain at the bend of the disk 2 significantly 6 increases the risk of disk failure during operation and 7 8 interferes with the distribution of material strain across the disk 2. In addition, the variation in the clamping strength of 9 the supporting structure 12 and the loss of material strength of 10 the disk 2 at the structure 12 makes it difficult to predict how 11 many cycles of operation the disk 2 can safely withstand. 12

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SUMMARY OF THE INVENTION

Accordingly, it is a general purpose and primary object of the present invention to provide an elastomeric disk resistant to cyclic material failure at its periphery.

It is a further object of the present invention to provide an elastomeric disk in which the peak material strain levels of the disk are located proximate to the center of the disk such that the cycles of operation for the disk can be adequately determined.

1 It is a still further object of the present invention to 2 provide an elastomeric disk which securely attaches to the 3 structure of an elastomeric ejection system.

To attain the objects described there is provided an elastomeric disk with a thickened curvature protruding from both sides of a central plane of the disk. Integral to each curvature of the disk is an annulet which dovetails from the curvatures to a periphery of the disk. The annulet allows secure attachment of the disk to the ejection system preferably by a clamp of the supporting structure of the ejection system.

In contrast to the prior art, the contour of the disk is 11 asymmetrical at the periphery of the disk. Specifically, the 12 thickness of the annulet in regard to the central plane is 13 greater on the pressure side of the disk. By increasing the 14 thickness of the annulet on the pressure side of the disk, the 15 bending strain and resultant material strain on the disk caused 16 by expansion is compensated for while the holding action of the 17 clamp is maintained. The strengthening of the disk thereby 18 lengthens the material cyclic life of the ejection system since 19 fatigue problems associated with the material strain at the 20 periphery bend are minimized. In addition, the reduction of 21 material strain at the periphery has the result of relocating 22 the higher material strain away from the clamp and towards the 23 center axis of the disk. At the center portion of the disk, 24

incidence of fatigue failure is generally expected and thus a
 fatigue failure becomes more predictable for maintenance
 scheduling.

The above and other features of the invention, including 4 various and novel details of construction and combinations of 5 parts will now be more particularly described with reference to 6 7 the accompanying drawings and pointed out in the claims. It will be understood that the particular devices embodying the 8 invention are shown by way of illustration only and not as the 9 limitations of the invention. The principles and features of 10 this invention may be employed in various and numerous 11 embodiments without departing from the scope of the invention. 12

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BRIEF DESCRIPTION OF THE DRAWINGS

A more complete understanding of the invention and many of the attendant advantages thereto will be readily appreciated as the same becomes better understood by reference to the following detailed description when considered in conjunction with the accompanying drawings wherein:

FIG. 1 depicts a plan view of a prior art elastomeric disk; FIG. 2 depicts a cross-sectional view of the prior art elastomeric disk in which the disk is clamped to the support structure of an elastomeric ejection system with the view of the disk taken from reference line 2-2 of FIG. 1;

FIG. 3 depicts a cross-sectional view of the prior art disk of FIG. 1 in which the disk is in an expanded state;

FIG. 4 depicts a plan view of an elastomeric disk of thepresent invention;

5 FIG. 5 depicts a cross-sectional view of the disk of the 6 present invention with the view taken from reference line 5-5 of 7 FIG. 4;

8 FIG. 6 depicts a cross-sectional view of the disk of the 9 present invention in which the disk is clamped to the support 10 structure of an elastomeric ejection system; and

FIG. 7 depicts a cross-sectional view of the disk of the present invention in which the disk is in an expanded state.

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DESCRIPTION OF THE PREFERRED EMBODIMENT

Referring now to the drawings including those drawings 15 provided in the prior art portion of this specification wherein 16 like numerals refer to like elements throughout the several 17 18 views, one sees that FIG. 4 depicts the elastomeric disk 20 of 19 the present invention. As shown in the cross-sectional view of FIG. 5, the elastomeric disk 20 is formed with a first curvature 20 22 protruding from a first side of a central plane 24 and a 21 second curvature 26 protruding from a second side of the central 22 plane 24. Integral to the curvatures 22 and 26 is an annulet 28 23

which dovetails from the taper of the curvatures 22, 26 to a
 periphery 30 of the disk 20.

As further shown in the figure, the contour of the disk 20 3 is asymmetrical to the central plane 24 at the annulet 28 of the 4 disk 20. In order to compensate for bending strain associated 5 with attachment to the structure of an ejection system, the 6 thickness of the annulet 28 on the pressure surface 32 is 7 increased. The portion of the annulet 28 on the pressure side 8 32 originates at a point 33 with the point 33 located a distance 9 10 "A" from a center axis 34 of the disk 20. The distance "A" is approximately eighty percent of the distance "B" for the point 11 35 upon which the annulet 28 originates on the non-pressure 12 surface 36 of the disk 20. By originating at the shorter 13 distance of "A", the annulet 28 incorporates a thicker area of 14 the first curvature 22. 15

In order to reduce the amount of material used while 16 maintaining an increased thickness of the annulet 28, the 17 pressure surface 32 of the annulet 28 indents toward the central 18 plane 24 without indenting the thickness of the annulet 28 19 20 between the origination points 33 and 35. The indent 37 is preferably positioned at a majority of the distance to the 21 periphery 30 from the origination point 33. For the remaining 22 distance to the periphery 30, the pressure surface 32 extends 23

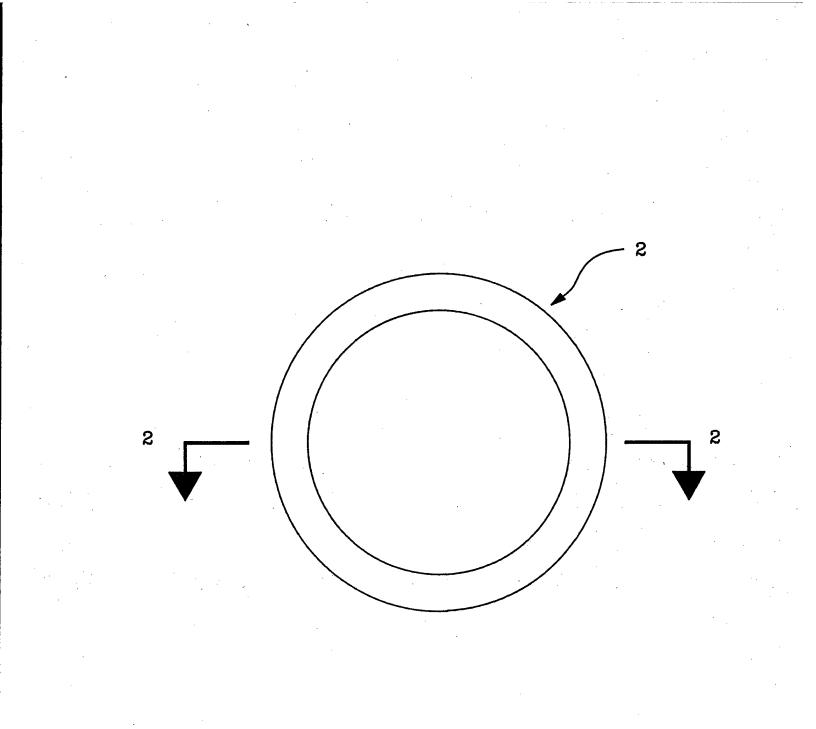
1 away from the central plane 24 to form the widened base 38 of 2 the annulet 28.

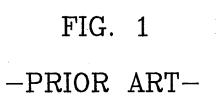
As shown in FIG. 6, the dovetailed shape of the annulet 28 3 still permits clamping by a clamp 39 at the contact area 40 4 while an arc of clamping by the clamp 39 is maintained on the 5 non-pressure surface 36 at the contact area 42. The result of 6 the increased thickness of the annulet 28 is that the bend area 7 44 shown in FIG. 7 diverts to the center axis 34 and the annulet 8 28 is also retained with minimal movement in the clamp 39. As 9 such, a shift of material strain continues to the central axis 10 34 of the disk 20. 11

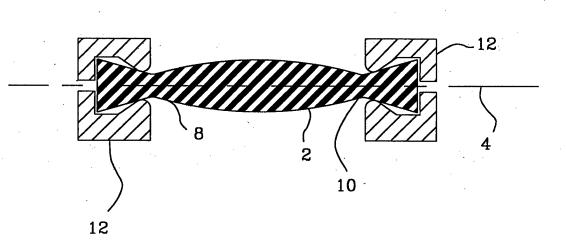
Accordingly, the shift of material strain to the center of the disk 20, also shifts the strain to where a fatigue failure is generally expected and thus a fatigue failure becomes more predictable for maintenance scheduling.

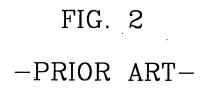
Thus by the present invention its objects and advantages are realized and although preferred embodiments have been disclosed and described in detail herein, its scope should be determined by that of the appended claims.

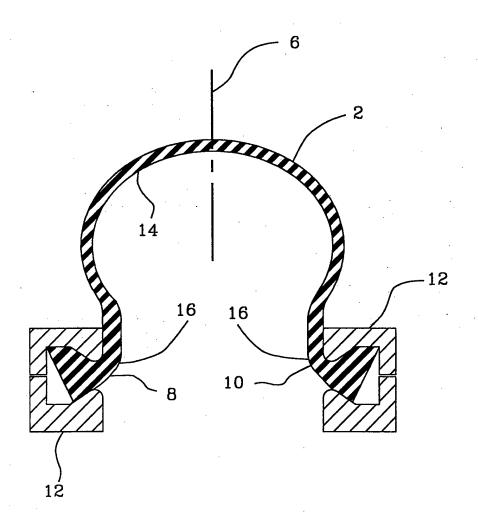
3	ASYMETRICALLY CONTOURED ELASTOMERIC DISK
4	ABSTRACT OF THE DISCLOSURE
5	An elastomeric disk for use in an elastomeric ejection
6	system in which a thickened curvature protrudes from both sides
7	of a central plane of the disk. Integral to each curvature is
8	an annulet which dovetails from the curvatures to a periphery of
9	the disk. The thickness of the annulet in regard to the central
10	plane is greater on the fluid pressure side of the disk thereby
11	shifting material strain to the center of the disk during
12	expanding deformation. The annulet shape is attachable to the
13	elastomeric ejection system by a clamp of the system.

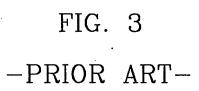


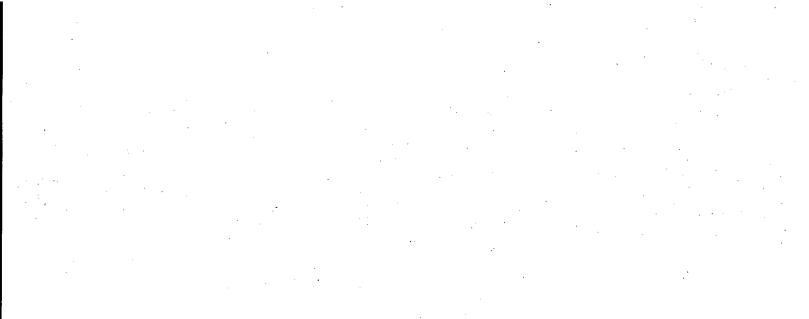












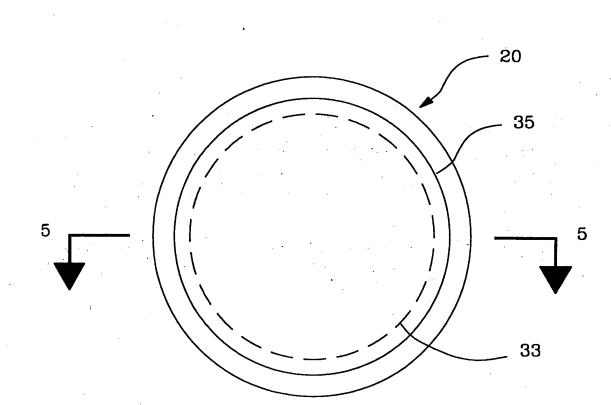
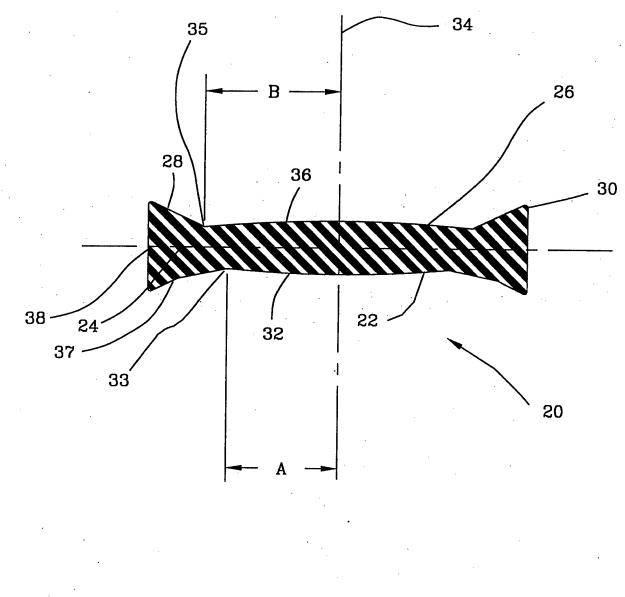


FIG. 4





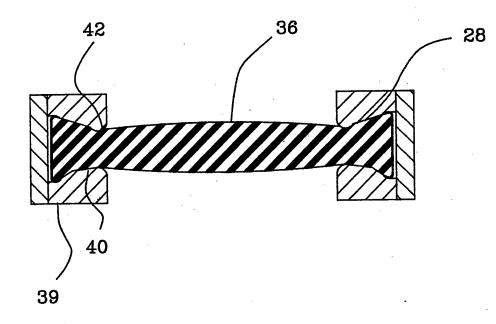


FIG. 6

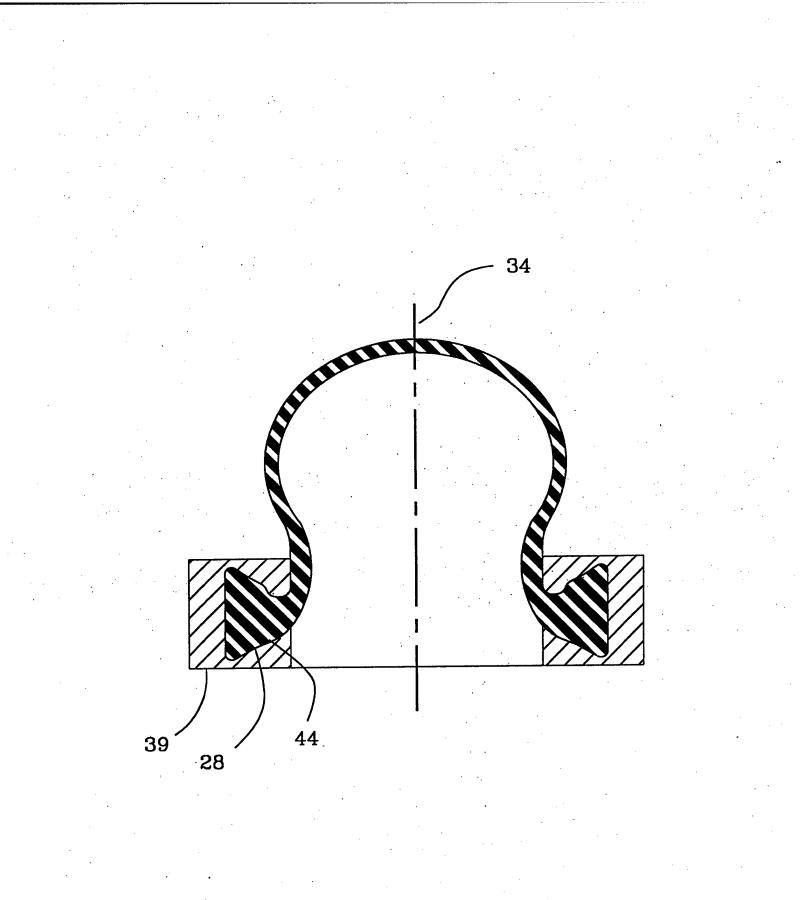


FIG. '