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TURBULENT BOUNDARY LAYER THICKNESS ESTIMATION  
METHOD AND APPARATUS

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TO ALL WHOM IT MAY CONCERN:

BE IT KNOWN THAT (1) WILLIAM L. KEITH and (2) KIMBERY M. CIPOLLA, citizens of the United States of America, employees of the United States Government, resident of (1) Ashaway, County of Washington, State of Rhode Island, and (2) Portsmouth, County of Newport, State of Rhode Island, have invented certain new and useful improvements entitled as set forth above of which the following is a specification:

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PATENT TRADEMARK OFFICE

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TURBULENT BOUNDARY LAYER THICKNESS ESTIMATION

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METHOD AND APPARATUS

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STATEMENT OF GOVERNMENT INTEREST

7 The invention described herein may be manufactured and used  
8 by or for the Government of the United States of America for  
9 governmental purposes without the payment of any royalties  
10 thereon or therefor.

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BACKGROUND OF THE INVENTION

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(1) Field of the Invention

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15 This invention generally relates to a technique for  
16 turbulent boundary layer thickness estimating using hot film wall  
17 shear stress sensors.

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21 More particularly, the invention relates to a technique for  
22 estimating turbulent boundary layer thickness in an underwater  
23 environment using hot film shear wall stress sensors and  
24 correlation coefficients.

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(2) Description of the Prior Art

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30 The art for hot wire anemometry has been widely used since  
31 the 1950's as a technique for making measurements of velocity and  
32 shear stress in experimental fluid mechanics facilities. Non-  
33 intrusive hot film sensors were developed in the late 1960's to

1 measure the wall shear or tangential stress. These sensors take  
2 advantage of the relationship between the rate of heat transfer  
3 from small thermal elements and the local wall shear stress. The  
4 wall shear stress is related to the velocity gradient at the wall  
5 by the relation:

$$6 \quad \tau = \mu \left. \frac{\partial u}{\partial y} \right|_{y=0} \quad (1)$$

7 Since the metal film used is adhered to a hard backing or  
8 substrate, the sensor is remarkably robust and useful for  
9 underwater applications. Whereas pressure sensors are typically  
10 used in both laboratory and real-world settings, hot film sensors  
11 have not been implemented as a diagnostic measurement tool on  
12 actual underwater or surface vehicles. This invention proposes  
13 to extend the range of applications to cases including at-sea  
14 testing and tactical operations. Currently, no non-intrusive  
15 measurement techniques exist for quantifying the turbulent  
16 boundary layer thickness outside of a laboratory environment.

17 The following patents, for example, disclose devices for  
18 detecting turbulent flow, but do not disclose the use of hot film  
19 sensors and correlation functions for measuring a turbulent  
20 boundary layer as disclosed in the present invention.

21 U.S. Patent No. 4,188,823 to Hood;

22 U.S. Patent No. 4,350,757 to Montag et al.;

23 U.S. Patent No. 4,774,835 to Holmes et al.;

24 U.S. Patent No. 4,993,261 to Lambert;

1 U.S. Patent No. 5,272,915 to Gelbach et al.; and

2 U.S. Patent No. 5,890,681 to Meng.

3 Specifically, Hood discloses a system for detecting the  
4 laminar to turbulent boundary layer transition on a surface while  
5 simultaneously taking pressure measurements. The system uses an  
6 accelerometer for producing electrical signals proportional to  
7 the noise levels along the surface and a transducer for producing  
8 electrical signals proportional to pressure along the surface.  
9 The signals generated by the accelerometer and transducer are  
10 sent to a data reduction system for interpretation and storage.

11 The patent to Montag et al. discloses a method for making  
12 visible by photochemical means residual moisture distributions in  
13 photographic wet film layers subjected to a gas flow. According  
14 to the invention, a film diffusely pre-exposed is immersed in an  
15 aqueous swelling agent solution which contains either (a) a  
16 reducing agent or (b) an alkali. After being exposed to the air  
17 stream, the invisible residual moisture profile is immersed in an  
18 alcoholic solution of either (a) an alkali or (b) a reducing  
19 agent. The half-tone image produced serves for determining  
20 stationary local boundary layer thickness distributions, wall  
21 shearing stresses, material transfer coefficients and heat  
22 transfer coefficients.

23 Holmes et al. discloses a method of visualizing laminar to  
24 turbulent boundary layer transition, shock location, and laminar  
25 separation bubbles around a test surface. A liquid crystal

1 coating is formulated using an unencapsulated liquid crystal  
2 operable in a temperature bandwidth compatible with the  
3 temperature environment around the test surface. The liquid  
4 crystal coating is applied to the test surface, which is  
5 preferably pre-treated by painting with a flat black paint to  
6 achieve a deep matted coating, after which the surface is  
7 subjected to a liquid or gas flow. Color change in the liquid  
8 crystal coating is produced in response to differences in  
9 relative shear stress within the boundary layer around the test  
10 surface.

11 Lambert discloses a fluid flow meter including a sensor  
12 mounted on or in the inner surface of a conduit for measuring  
13 fluid flow through the conduit where the sensitivity of the  
14 sensor is dependent upon the thickness of the fluid boundary  
15 layer extending over the sensor. According to the invention,  
16 fluid is drawn out of the conduit through an aperture located a  
17 predetermined distance upstream of the sensor to remove the  
18 boundary layer developed upstream of the sensor thereby rendering  
19 the sensor immune to fluctuations in the thickness of the removed  
20 boundary layer. At the same time, a fresh boundary layer of  
21 reduced thickness and greater stability is initiated over the  
22 sensor so as to improve the sensitivity and repeatability of the  
23 sensor.

24 The patent to Gelbach et al. discloses an airflow sensing  
25 system for determining the type of airflow flowing over a flight

1 surface. A hot film sensor is driven by a constant voltage  
2 feedback circuit that maintains the voltage across the sensor at  
3 a predetermined level. A signal processing circuit receives an  
4 output signal of the feedback circuit and determines whether the  
5 output signal is indicative of laminar, transitional, or  
6 turbulent airflow. The transitional airflow is distinguished  
7 from turbulent airflow by a signal having significant energy in a  
8 low-frequency pass band from 50-80 Hz. The signal processing  
9 circuit drives a three-color LED display to provide a visual  
10 indication of the type of airflow being sensed.

11 Meng discloses a method for controlling microturbulence in a  
12 medium flowing near a surface. The method includes the steps of  
13 measuring the forces acting near or on the surface and using  
14 those measurements to determine the state probabilities for the  
15 microturbulent events occurring at the surface. The control  
16 method then activates selective cells in an array of cells to  
17 apply forces at the surface to counteract the microturbulent  
18 events and thus reduce turbulence. Each cell has a pair of  
19 electrodes and opposing magnetic poles such that when the control  
20 method activates a cell, the interaction of the electric field  
21 and the magnetic field at the cell creates a Lorentz force normal  
22 to the surface.

23 It should be understood that the present invention would in  
24 fact enhance the functionality of the above patents by providing  
25 a unique concept for estimating the thickness of a hydrodynamic

1 turbulent boundary layer on undersea vehicles, surface vessels,  
2 towed bodies or in a laboratory setting. It utilizes  
3 commercially available hot film sensors to non-intrusively  
4 measure the thickness of the turbulent boundary layer on a  
5 surface.

6

7

#### SUMMARY OF THE INVENTION

8

Therefore it is an object of this invention to provide a  
9 method for measuring a turbulent boundary layer thickness.

10

Another object of this invention is to provide a method for  
11 measuring a turbulent boundary layer thickness using hot film  
12 wall shear stress sensors.

13

Still another object of this invention is to provide a  
14 method for measuring a turbulent boundary layer thickness  
15 utilizing sensor measurements and correlation coefficients.

16

A still further object of the invention is to provide a  
17 method for measuring turbulent boundary layer thickness in  
18 underwater applications.

19

In accordance with one aspect of this invention, there is  
20 provided a method and apparatus for determining turbulent  
21 boundary layer thickness. Specifically, a pair of sensors are  
22 mounted to a solid surface interfacing with a fluid at two  
23 separate stream wise locations. A voltage output from the pair  
24 of sensors is recorded and a real non-dimensional value of a  
25 correlation coefficient is computed with measured data from the

1 recorded voltage. A laboratory non-dimensional value of the  
2 correlation coefficient is independently determined from  
3 laboratory data. The real non-dimensional value is compared with  
4 the laboratory non-dimensional value to obtain a boundary layer  
5 thickness having a value which minimizes a difference between the  
6 values of the real non-dimensional value and the laboratory non-  
7 dimensional value.

#### 8 9 BRIEF DESCRIPTION OF THE DRAWINGS

10 The appended claims particularly point out and distinctly  
11 claim the subject matter of this invention. The various objects,  
12 advantages and novel features of this invention will be more  
13 fully apparent from a reading of the following detailed  
14 description in conjunction with the accompanying drawings in  
15 which like reference numerals refer to like parts, and in which:

16 FIG. 1 is a top schematic view of a typical configuration of  
17 flush mounted hot film sensors according to the present  
18 invention; and

19 FIG. 2 is a side schematic view of the configuration shown  
20 in FIG. 1.

#### 21 22 DESCRIPTION OF THE PREFERRED EMBODIMENT

23 A typical configuration for a relevant application to  
24 measure a turbulent boundary layer thickness is shown in FIGS. 1  
25 and 2. In particular, two hot film wall shear stress sensors 10

1 are housed in a single unit 12 which is mounted flush with a  
2 fluid-solid interface 14. A vessel speed is characterized by the  
3 label  $U_0$  and the vessel direction is shown by arrow 16. Multiple  
4 sensor units 12 can be positioned at various locations on a  
5 submarine hull or control surfaces (not shown) where boundary  
6 layer thickness and/or an indication of separation is of  
7 interest.

8 For example, when positioning a sensor unit 12 at the  
9 location of a hull array, a quantitative measure of the boundary  
10 layer thickness may be used to optimize the sonar design. Also,  
11 monitoring of the sensors 10 for anomalous readings, in  
12 conjunction with the acoustic array data, provides a means to  
13 determine the source of background noise in the sonar system.  
14 Specifically, upstream vortex shedding would be detected by the  
15 wall shear stress sensors, while structural vibrations would be  
16 detected by the sonar system only. On a control surface,  
17 monitoring for the onset of separation can be used to define the  
18 operating envelope for quiet, efficient maneuvering. This type  
19 of data could then be incorporated into an active control system.

20 FIG. 2 shows a side view of sensors 10A and 10B and an  
21 apparatus for estimating boundary layer thickness  $\delta$ . Sensor 10A  
22 is joined to a first analog to digital converter 18A to provide  
23 an analog shear stress measurement. Likewise, sensor 10B is  
24 joined to a second analog to digital converter 18B. First and  
25 second analog to digital converters 18A and 18B can be a single

1 analog to digital converter having multiple channels. A computer  
2 20 receives digital signals from first and second analog to  
3 digital converters 18A and 18B. The computer 20 is also joined  
4 to a memory element 22 having tabulated correlation coefficient  
5 values stored therein. The computer 20 can thus receive shear  
6 stress measurements and use them to compute a real correlation  
7 coefficient. The computer 20 can then compare the real  
8 correlation coefficient against tabulated correlation coefficient  
9 values stored in memory element 22 in order to provide an  
10 estimate of the boundary layer thickness.

11 Since hot film sensors have a finite area, spatial averaging  
12 over the sensor leads to attenuation in the frequency spectra.  
13 Unfortunately, the lack of experimental data makes it impossible  
14 to quantify this attenuation. Therefore, this invention proposes  
15 to use the correlation coefficient (also referred to as the  
16 normalized correlation function) as the metric to eliminate the  
17 effects of spatial averaging over the frequency range where  
18 adequate signal-to-noise exists. The correlation coefficient  
19  $R_{\tau_1\tau_2}(\xi_n, T)$  is defined as:

$$20 \quad R_{\tau_1\tau_2}(\xi_n, T) = \frac{\langle \tau_1(x, t) \tau_2(x + \xi_n, t + T) \rangle}{\sqrt{\langle \tau_1(x, t)^2 \rangle} \sqrt{\langle \tau_2(x + \xi_n, t + T)^2 \rangle}} \quad (2)$$

21 where  $\tau_1$  and  $\tau_2$  are the wall shear stress values at two separate  
22 stream wise locations,  $x$  is the stream wise coordinate,  $\xi_n$  is the  
23 discrete sensor spacing, and the  $\langle \rangle$  indicate temporal mean

1 quantities. The estimation of  $R_{r_1 r_2}(\xi_n, T)$  is determined digitally  
2 in practice, and involves modern analog-to-digital converters and  
3 computers. A non-dimensional form of the correlation  
4 coefficient,  $\hat{R}_{r_1 r_2}(\hat{\xi}_n, \hat{T})$  is obtained by defining  $\xi_n = \hat{\xi}_n \times \delta$  and  
5  $T = \hat{T} \delta / U_0$ , where  $\delta$  is the turbulent boundary thickness and  $U_0$  is  
6 the ship speed or free stream velocity. This leads to a direct  
7 relation between the correlation coefficient,  $\hat{R}_{r_1 r_2}(\hat{\xi}_n, \hat{T})$  and the  
8 turbulent boundary thickness  $\delta$ .

9 Knowledge of the turbulent boundary thickness  $\delta$  is of  
10 particular interest to designers of sonar systems, including hull  
11 mounted sonar and towed arrays. Sonar systems must be designed  
12 to filter unwanted non-acoustic noise resulting from turbulent  
13 boundary layer fluctuations, in order to maximize detection and  
14 classification. Design parameters include the structural  
15 configuration and the geometry of the sensors themselves. In  
16 addition, in-situ measurements of the mean wall shear stress can  
17 be used to quantify the skin friction of submarines, unmanned  
18 undersea vehicles (UUVs), surface vessels or towed bodies under  
19 operating conditions. Reduction of skin friction drag is also of  
20 primary interest to the design of racing yachts and high-speed  
21 intercept vessels. This is evidenced by the use of riblets on  
22 the surface of America's Cup yachts. Detailed measurements of  
23 one or more boundary layer parameters on full-scale hulls would

1 provide quantitative information necessary to improve the design.  
2 Finally, boundary layer separation resulting from vehicle  
3 maneuvers is a concern because it leads to a significant increase  
4 in the overall drag of the body. This separation is preceded by  
5 an increase in the boundary layer thickness and decrease in the  
6 mean wall shear stress, both of which can be detected by the  
7 proposed sensors and methodology of the present invention.

8 An inherent problem with the commercially available  
9 technology is the difficulty in calibrating the sensors.  
10 Calibration of a single sensor required simultaneous measurements  
11 of the mean voltage output from the sensor and the mean velocity  
12 profile at the sensor location, from which the mean wall shear  
13 stress is calculated. The result of a typical calibration is a  
14 polynomial relationship between the wall shear stress,  $\tau$  and the  
15 voltage output,  $V$  from the sensor, such as,  $\tau = aV^3 + bV^2 + cV + d$ . The  
16 calibration parameter  $d\bar{\tau}/d\bar{V}$ , is determined from the slope of  
17 the calibration curve and used to convert sensor output voltage  
18 to wall shear stress. For a complete calibration, the mean  
19 velocity and the fluid temperature must be systematically varied,  
20 and the measurements repeated. Additional complications include  
21 the thermal response of the sensor, spatial averaging due to the  
22 sensor size and the non-linearity of the relationship between  
23 voltage and mean wall shear stress. For these reasons, the use  
24 for flush mounted wall shear stress sensors in laboratory or  
25 actual applications has been limited. However, the technique

1 described here uses the normalized correlation coefficient as  
2 defined in equation (2). Since the measured root mean square  
3 values for each sensor are used to normalize the fluctuation  
4 signal in this expression, the calibration parameters cancel.

5 Thus, the primary purpose of the present technique for  
6 turbulent boundary layer thickness estimation using hot film wall  
7 shear stress sensors is to obtain an estimation of the turbulent  
8 boundary layer thickness in underwater applications. This  
9 quantity is often difficult or impossible to measure due to  
10 technical limitations of conventional techniques. The intent is  
11 to provide an inexpensive method for boundary layer thickness  
12 estimation utilizing existing commercially available technology.  
13 Conventional laboratory methods such as laser Doppler velocimetry  
14 (LDV), hot-wire traverses and pitot tubes are all impractical for  
15 actual applications due to physical constraints and the potential  
16 for damaging the instruments. Two metrics which could be used  
17 are fluctuating wall pressure from piezoelectric sensors and wall  
18 shear stress from flush mounted hot film sensors. Both  
19 quantities result from velocity fluctuations in the inner and  
20 outer regions of the turbulent boundary layer. However, wall  
21 shear stress measurements are directly related to the velocity  
22 gradient near the wall, while pressure fluctuations measured at  
23 the wall are due to both incompressible velocity fluctuations  
24 (non-acoustic) in the boundary layer and structural vibrations  
25 and acoustic waves in the water. Consequently, any statistical

1 parameter from wall pressure measurements will contain  
2 contributions from acoustic and structural sources, which cannot  
3 be distinguished from turbulent velocity fluctuation  
4 contributions for a given sensor. While an array of wall  
5 pressure sensors could be used to distinguish these sources, the  
6 present invention instead develops a low-cost system containing a  
7 minimum number of sensors and related signal processing.  
8 Therefore, the proposed technique uses the wall shear stress as  
9 the metric of interest.

10 The principal of operation is as follows. The voltage from  
11 a pair of sensors is recorded and  $R_{r_1 r_2}(\xi_n, T)$  is computed from  
12 equation (2). The boundary layer thickness is treated as an  
13 unknown parameter and used to obtain a non-dimensional value for  
14  $R_{r_1 r_2}(\xi_n, T)$ . To determine its value, the value of  $\hat{R}_{r_1 r_2}(\hat{\xi}_n, \hat{T})$   
15 calculated from the measured data is compared to tabulated values  
16 of  $\hat{R}_{r_1 r_2}(\hat{\xi}_n, \hat{T})$  obtained in the laboratory. The boundary layer  
17 thickness is determined as the value that minimizes the  
18 differences between these values. While the sensor is highly  
19 sensitive to the temperature of the working fluid, this effect is  
20 eliminated by considering a normalized quantity. Measurements of  
21 the cross spectral characteristics of wall shear stress were  
22 first reported in 1997 and provided quantitative information  
23 regarding the convection of shear stress producing structures in  
24 the boundary layer.

1           The advantages of the present invention are numerous. Since  
2 the sensors are extremely small and compact, the system can be  
3 mounted in experimental facilities or on control surfaces where  
4 hot wire probes or pitot tubes are too large and intrusive.  
5 Further, the invention utilizes a minimum number of commercially  
6 available, inexpensive sensors, commercially available anemometry  
7 and minimal processing using PC-based algorithms. The invention  
8 has been designed to be compatible with existing laboratory and  
9 non-laboratory systems and therefore can be easily installed in  
10 any underwater application. Since the methodology utilizes a  
11 normalized quantity, the need for a complete calibration of each  
12 individual sensor is eliminated. Hot film sensors are sturdier  
13 and less prone to fouling than hot wire sensors. Therefore, this  
14 system and the associated sensors require only minimal  
15 maintenance. The invention is modular and easy to transport,  
16 does not require extensive training or safety procedures for the  
17 operator, and is durable with no protruding parts that would be  
18 easily broken, making them ideal for underwater applications.

19           In view of the above detailed description, it is anticipated  
20 that the invention herein will have far reaching applications  
21 other than those of determining underwater turbulent boundary  
22 layer thickness.

23           This invention has been disclosed in terms of certain  
24 embodiments. It will be apparent that many modifications can be  
25 made to the disclosed apparatus without departing from the

1 invention. Therefore, it is the intent of the appended claims to  
2 cover all such variations and modifications as come within the  
3 true spirit and scope of this invention.

2  
3 TURBULENT BOUNDARY LAYER THICKNESS ESTIMATION

4 METHOD AND APPARATUS

5  
6 ABSTRACT OF THE DISCLOSURE

7 A method and apparatus are presented for determining  
8 turbulent boundary layer thickness. In this method and  
9 apparatus, a pair of sensors are mounted to a solid surface  
10 interfacing with a fluid at two separate stream wise locations.  
11 A voltage output from the pair of sensors is recorded and a real  
12 non-dimensional value of a correlation coefficient is computed  
13 with measured data from the recorded voltage. A laboratory non-  
14 dimensional value of the correlation coefficient is independently  
15 determined from laboratory data. The real non-dimensional value  
16 is compared with the laboratory non-dimensional value to obtain a  
17 boundary layer thickness having a value which minimizes a  
18 difference between the values of the real non-dimensional value  
19 and the laboratory non-dimensional value.