

VORTICITY GENERATOR FOR IMPROVING HEAT EXCHANGER EFFICIENCY

TO ALL WHOM IT MAY CONCERN:

BE IT KNOWN THAT (1) WILLIAM L. KEITH and (2) KIMBERLY M. CIPOLLA, citizens of the United States of America, employees of the United States Government, and residents of (1) Ashaway, County of Washington, State of Rhode Island, and (2) Portsmouth, County of Newport, State of Rhode Island, have invented certain new and useful improvements entitled as set forth above, of which the following is a specification.

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IN REPLY REFER TO:

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The below identified patent application is available for licensing. Requests for information should be addressed to:

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1 Attorney Docket No. 83308

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3 VORTICITY GENERATOR FOR IMPROVING HEAT EXCHANGER EFFICIENCY

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5 STATEMENT OF GOVERNMENT INTEREST

6 The invention described herein may be manufactured and used
7 by or for the Government of the United States of America for
8 Governmental purposes without the payment of any royalties
9 thereon or therefor.

10

11 CROSS REFERENCE TO OTHER PATENT APPLICATIONS

12 Not applicable.

13

14 BACKGROUND OF THE INVENTION

15 (1) Field of the Invention

16 The invention relates to heat exchangers and is directed
17 more particularly to an improvement which renders current heat
18 exchangers more efficient.

19 (2) Description of the Prior Art

20 Conventional heat exchangers typically include a tube for
21 flowing a working fluid therethrough, the tube passing through or
22 proximate a thermal medium, hot or cold, to heat or cool the
23 working fluid. The thermal medium may itself be a flowing fluid.

24 In an effort to improve heat transfer from the thermal
25 medium to the working fluid, in some instances, the outer surface

1 of the tube is increased by the use of external fins or the like.
2 In other instances, a structure is placed in the tube to generate
3 vorticity or turbulence within the tube to increase heat
4 exchange.

5 For example, in U.S. Patent No. 4,062,524, issued Dec. 13,
6 1999 to Dieter Brauner et al, there is disclosed an arrangement
7 of comb-like plates for static mixing of fluids. In U.S. Patent
8 No. 4,208,136, issued June 17, 1980 to Leonard T. King, there is
9 shown a tube with mixing elements therein, the elements being
10 shaped to impart a rotational vector to portions of the flow
11 stream. In U.S. Patents Nos. 4,466,741 and 5,312,185, issued
12 August 21, 1984 and May 17, 1994, respectively, to Hisao Kojima,
13 there are shown arrangements of helical blades mounted in tubes.
14 In U.S. Patent No. 5,518,311, issued May 21, 1996, to Rolf
15 Althaus et al, and in U.S. Patent No. 5,803,602, issued September
16 8, 1998, to Adnan Eroglu et al, there are shown triangularly-
17 shaped vortex generators mounted in flow ducts.

18 The above-noted prior art devices have inherent
19 disadvantages, including pressure drop through the heat
20 exchanger. Helical designs and structures extending width-wise
21 of the tube require increased power input to compensate for
22 friction losses. Further, the relatively large volume of some of
23 the above-noted mixing elements consume much of the cross-section
24 of the tube, reducing the volume available for fluid flow. Still

1 further, the relatively large volume devices result in much
2 heavier tubes.

3 Accordingly, there is a need for an improved heat exchange
4 tube and system in which heat transfer within a tube conveying a
5 working fluid is substantially enhanced, without adding a
6 substantial volume of blocking structure in the tube or
7 significant weight to the tube, and which does not cause a
8 meaningful pressure drop in the tube, or require further power
9 input to force the fluid therethrough.

10

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SUMMARY OF THE INVENTION

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13 An object of the invention is, therefore, to provide a heat
14 exchange tube featuring structure therein which improves heat
15 transfer, but does not occupy a substantial volume of the tube
16 nor add substantial weight to the tube.

17

18 A further object of the invention is to provide such a heat
19 exchange tube in which the internally-mounted structure does not
20 precipitate a meaningful pressure drop in the tube and does not
21 increase resistance to flow such as to require added power to
22 flow the working fluid therethrough.

23

24 A further object of the invention is to provide a heat
25 exchange system featuring a tube as described immediately above.

26

27 A still further object of the invention is to provide a
28 method for improving heat exchangers, including conventional heat
29 exchangers.

1 With the above and other objects in view, a feature of the
2 present invention is the provision of a heat exchange tube
3 comprising a tubular conduit for flowing a working fluid
4 therethrough and for conducting heat between the working fluid
5 and a thermal field proximate to the tube, and a wire extending
6 axially through the tube and spaced from an inside surface of the
7 tube.

8 In accordance with a further feature of the invention, there
9 is provided a heat exchange system including a thermal source
10 providing a fluid heat exchange medium, a heat exchanger for
11 receiving the heat exchange medium, a heat exchange tube
12 extending through the heat exchanger and adapted to flow working
13 fluid therethrough, and a wire extending axially through the tube
14 and spaced from an inside surface of the tube.

15 In accordance with a still further feature of the invention,
16 there is provided a method for improving heat exchange capacity
17 in a heat exchange tube including a tubular conduit for flowing a
18 working fluid therethrough and for conducting heat between the
19 working fluid and a thermal field proximate the tube, the method
20 comprising providing a wire in the tube extending axially of the
21 tube and spaced from an inside surface of the tube.

22 The above and other features of the invention, including
23 various novel details of construction and combinations of parts
24 and method steps, will now be more particularly described with
25 reference to the accompanying drawings and pointed out in the

1 claims. It will be understood that the particular devices and
2 method embodying the invention are shown by way of illustration
3 only and not as limitations of the invention. The principles and
4 features of this invention may be employed in various and
5 numerous embodiments without departing from the scope of the
6 invention.

7

8

BRIEF DESCRIPTION OF THE DRAWINGS

9 Reference is made to the accompanying drawings in which are
10 shown illustrative embodiments of the invention, from which its
11 novel features and advantages will be apparent, and wherein
12 corresponding reference characters indicate corresponding parts
13 throughout the several views of the drawings and wherein:

14 FIG. 1 is a diagrammatic sectional view of one form of heat
15 exchange tube illustrative of an embodiment of the invention;

16 FIG. 2 is a diagrammatic sectional view of the tube of FIG.
17 1 shown in a heat exchanger;

18 FIG. 3 is a sectional view taken along line III-III of FIG.
19 2;

20 FIG. 4 is a sectional view similar to FIG. 3, but
21 illustrative of an alternative embodiment;

22 FIGS. 5 and 5A are similar to FIG. 4, but illustrative of
23 further alternative embodiments; and

1 The wire may be of metal, or temperature resistant plastic,
2 or a composite thereof. The wire 32 is quite thin, in the range
3 of about .01-.1 inch diameter and preferably in the range of
4 about .02-.04 inch diameter, depending in large measure on the
5 diameter of the conduit inside surface 34. If a single wire is
6 used, it preferably is located substantially centrally of the
7 tube 20 and extends axially thereof. However, for specific
8 applications the wire may be placed off-center.

9 Turbulent flow generation has been experimentally observed
10 using a wire having a diameter as large as .08 inches and as
11 small as .02 inches in a 2 inch diameter conduit. The largest
12 wire observed reduces the cross sectional area of the conduit by
13 only .16%. The effect is expected to be useful for wires
14 occupying as much as 1% of the cross sectional area of a conduit.
15 The lower end of the effect is unknown, but based on the above
16 observations, it extends to wires occupying as little as .01% of
17 the cross sectional area of the conduit. Based on the teachings
18 of Incropera and DeWitt, Fundamentals of Heat and Mass Transfer,
19 2d Edition, at page 399-400, where they discuss flow through a
20 concentric tube annulus, one would not expect a wire having this
21 small a cross sectional area to significantly affect turbulent
22 flow. Thus, the generation of turbulent flow by a member having
23 such a small cross sectional area is unexpected in view of the
24 prior art.

1 The wire 32 may be mounted by any manner not in
2 contravention of the objects of the invention, that is in any
3 manner not imposing substantial blockage, weight, pressure drops,
4 a need for increased power, and the like. In a preferred
5 embodiment, the wire is fixed to the tops of thin rigid posts 46
6 extending inwardly from the conduit inside surface 34 (two shown
7 in FIG. 1).

8 Referring to FIGS. 4 and 5, it will be seen that additional
9 wires 40, 42 (FIG. 4), 44 (FIG. 5) may be used. In such
10 instances, all the wires preferably are spaced from the tube
11 inside surface 34 equidistantly, and spaced from each other. In
12 FIG. 5A there is shown an embodiment for providing maximum
13 turbulence in the boundary layer area of the conduit 22. The
14 wires are disposed in circular fashion around the axis of the
15 conduit and proximate the conduit inside surface 34.

16 Referring to FIG. 6, it will be seen that an illustrative
17 heat exchange system may include the thermal source 30 which may
18 be either a heat source or a cold source, or a combination
19 thereof. The heat exchange medium 28 flows from the thermal
20 source 30 to a heat exchanger 50 to establish the thermal field
21 26. The tube 20, carrying the working fluid 24, winds through
22 the heat exchanger 50 and the thermal field 26. The working
23 fluid 24 flows with increased vorticity and mixing, resulting
24 from the boundary layer on the wire, and its interaction with the
25 boundary layer on the tube wall.

1 While the tube and wire structure may be easily manufactured
2 for new equipment, the wire 32, or any selected number of wires,
3 can be retrofitted into existing heat exchange tubes rather
4 inexpensively and in short time spans.

5 There is thus provided a heat exchange tube and system which
6 provide for increased heat transfer while not presenting problems
7 related to blockage, weight, pressure drops, or a need for
8 additional power. Further, the invention provides a method for
9 improving the performance of conventional heat exchange tubes,
10 and thereby heat exchange systems.

11 It will be understood that many additional changes in the
12 details, materials, steps and arrangement of parts, which have
13 been herein described and illustrated in order to explain the
14 nature of the invention, may be made by those skilled in the art
15 within the principles and scope of the invention as expressed in
16 the appended claims.

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5 ABSTRACT OF THE DISCLOSURE

6 A heat exchange tube includes a tubular conduit for flowing
7 a working fluid therethrough and for conducting heat between the
8 working fluid and a thermal field proximate the tube, and a wire
9 extending axially through the tubular conduit and spaced from an
10 inside surface of the tubular conduit. The invention also
11 provides a method for increasing heat transfer about a tubular
12 conduit by positioning a wire in the conduit.

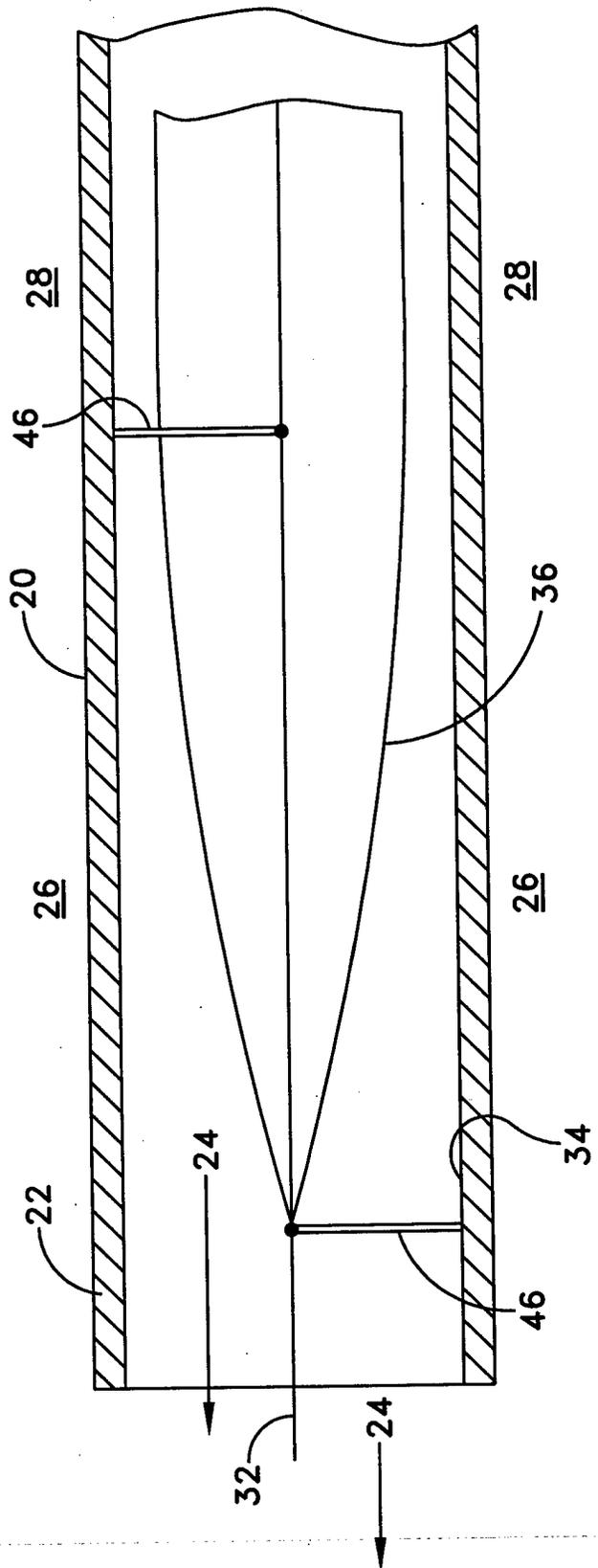


FIG. 1

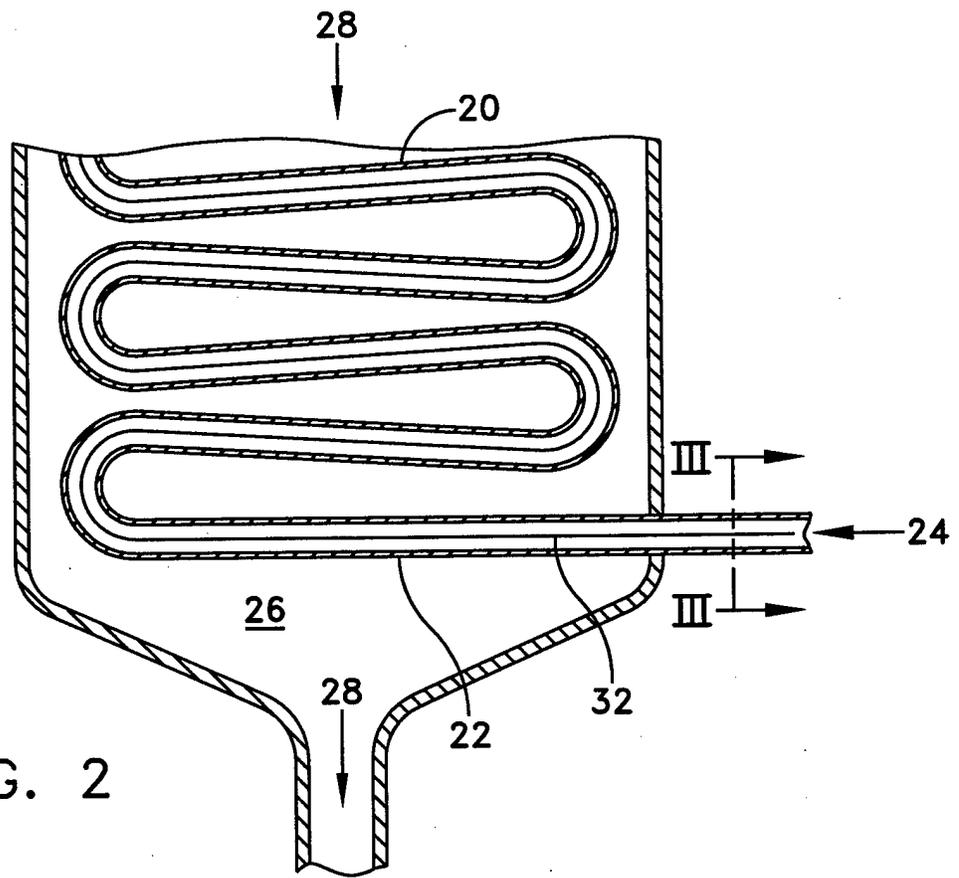


FIG. 2

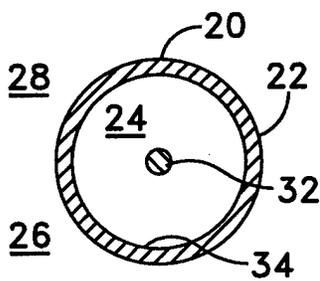


FIG. 3

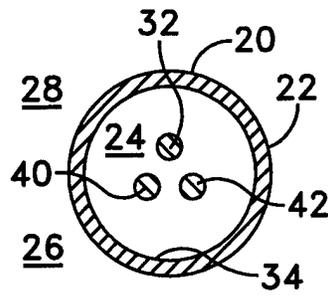


FIG. 4

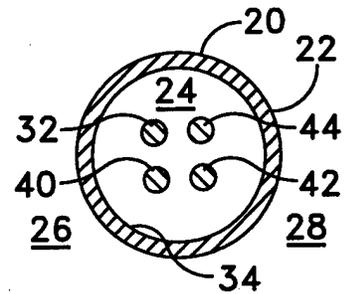


FIG. 5

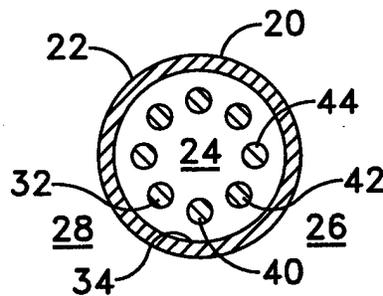


FIG. 5A

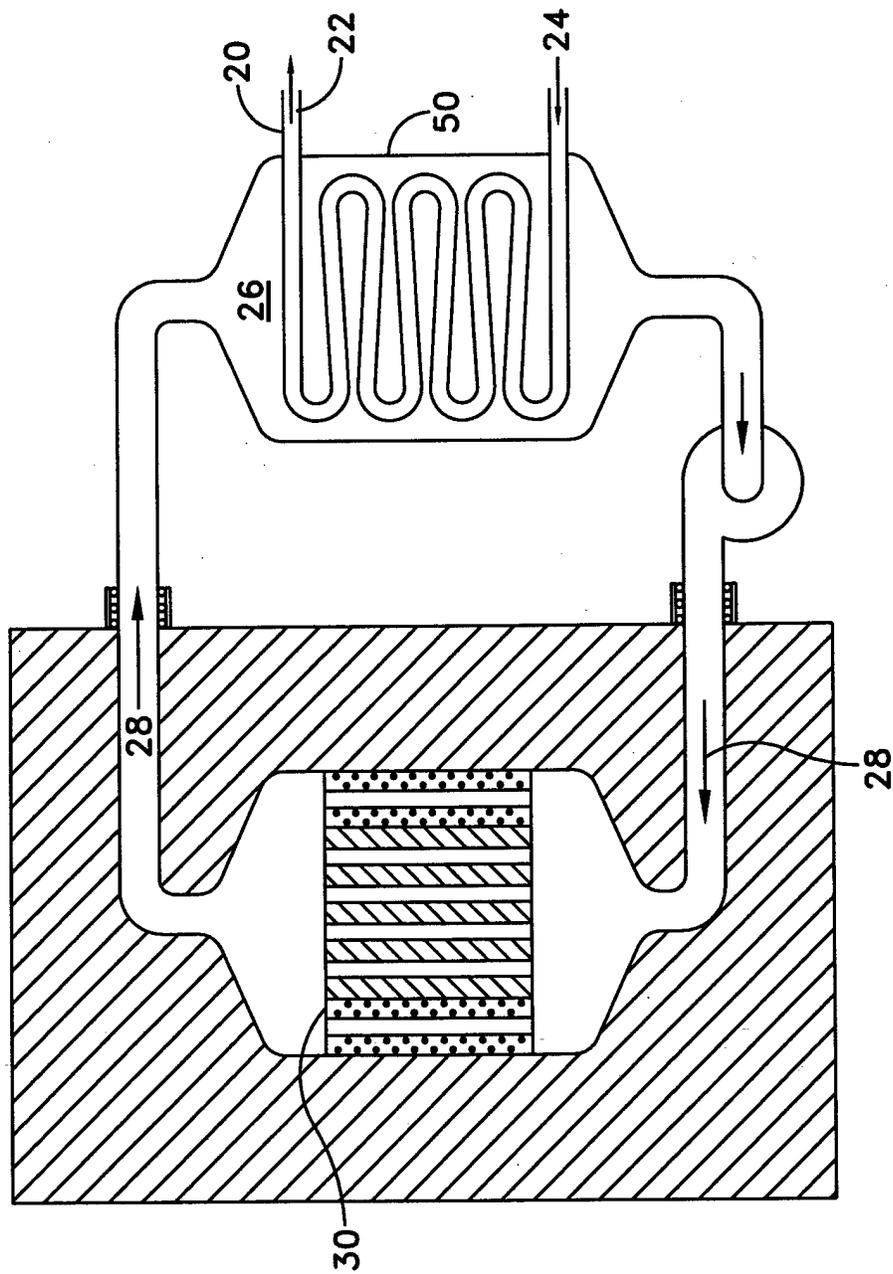


FIG. 6