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1   Attorney Docket No. 78471

2

3   COMPUTER IMPLEMENTED SYSTEM AND METHOD

4   FOR EVALUATING GAS GENERATOR LAUNCHERS

5

6   STATEMENT OF GOVERNMENT INTEREST

7   The invention described herein may be manufactured and used

8   by or for the Government of the United States of America for

9   governmental purposes without the payment of any royalties

10 thereon or therefore.

11

12   BACKGROUND OF THE INVENTION

13   (1) Field of the Invention

14   The present invention relates to a system and method for

15 modeling gas generator launchers, and more particularly to a

16 computer-implemented system and method of modeling a device

17 launched by way of a boosted argon hybrid (BAH) energy source.

18   (2) Description of the Prior Art

19   When properly designed and implemented, a BAH energy source

20 can be utilized as a device launcher. One such application of a

21 device launcher would be from a submerged vehicle, such as a

22 submarine, which may have a need to launch devices into the

1

1   surrounding environment. However, since the surrounding

2   environment is at a pressure higher than the interior pressure of

3   a submarine, an elevated pressurized launch system is needed.

4   One such device launcher is a boosted argon hybrid (BAH)

5   powered gas generator launcher. FIG. 1 shows a typical BAH

6   system with two BAH units 13 and a ram plate 30. During the

7   launch process fuel 22 converts into a hot pressurized gas in the

8   combustion chamber 16, and then flows into a gas expansion tank

9   18. The pressurized argon then flows through an external orifice

10   26 into a plenum chamber 24. The gasses expand in the plenum

11   chamber 24, and accelerate the ram plate 30 (if included) and the

12   device being launched 28. Gasses may flow around any gap 25

13   about the ram plate and device.

14   One approach to evaluating gas generator launcher designs is

15   to fabricate and actuate test gas generator launchers and to

16   physically measure their performance. As can be appreciated,

17   such fabrication and testing requires construction of the test

18   launcher, fabrication of gas generant fuel and measurement of

19   launcher perimeters, including pressures, temperatures,

20   velocities and acceleration during the actual firing of such a

21   launcher.

1   This type of testing is labor intensive, time consuming and

2   costly. Physical testing is also expensive because chemical

3   compounds, containers and measuring equipment employed are

4   subject to wear or destruction during each test. In order to

5   test launchers having different dimensions, separate test

6   apparatus must be constructed. Moreover, it would be difficult

7   to vary one or more launcher parameters in order to satisfy the

8   design goals should a specific launcher design fail to satisfy

9   performance goals. Accordingly, parameters that can be modified

10   are limited to fuel composition, geometry and quantity.

11   Thus, it would be an advancement in the art to provide a

12   system and method of modeling a device launcher which reduces or

13   eliminates the need for fabrication and testing. It is also an

14   advancement to provide such a system and method in a computerized

15   implementation which can be used to predict gas generator

16   performance characteristics without destructive testing of gas

17   generant compounds.

18

19   SUMMARY OF THE INVENTION

20   The present invention provides a computer-implemented system

21   and method for evaluating gas generator launchers, which include

22   at least one gas generator having a gas generant fuel housed

3

1 within a combustion chamber, a tank containing a compressed gas

2 in fluid communication with the combustion chamber through an

3 internal orifice. The internal orifice is sealed with a rupture

4 disk prior to gas generant ignition. The tank is also in fluid

5 communication with a plenum chamber through an external orifice.

6 The external orifice is also sealed with a second rupture disk

7 until the pressure in the compressed gas tank exceeds the design

8 characteristics of the rupture disk. Within the plenum chamber

9 is a device to be launched and, in certain embodiments, a ram

10 plate positioned intermediate the device and the external

11 orifice. The ram plate reduces losses resulting from the blow-by

12 of gasses around the device being launched.

13   The method includes fixing the system geometry and

14 establishing initial conditions of the launcher to be evaluated.

15 The initial conditions include a mass, composition and geometry

16 of the fuel included in the gas generant, geometries of the tank

17 internal and external orifices and plenum chamber, initial

18 pressures within the pressurized gas tank and plenum chamber, and

19 the mass and geometry of the device to be launched. In addition,

20 if included, the mass of the ram plate must be fixed. Otherwise,

21 the geometry of any gap existing between the device to be

4

1   launched and the walls of the plenum chamber must be factored

2   into the equation.

3   Next, a gas generator internal ballistics burn rate is

4   modeled, and an amount of mass and energy added to the combustion

5   chamber as the fuel is consumed is calculated. Then, the rate of

6   mass flux out of the combustion chamber and into the gas tank is

7   modeled utilizing internal orifice geometry, tank and combustion

8   chamber temperatures, tank gas mass, and a ratio of combustion

9   chamber and tank pressure.

10   A rate of energy flux out of the combustion chamber and into

11   the tank is also modeled using the rate of mass flux out of the

12   combustion chamber and into the tank, the combustion chamber

13   temperature and the specific heat of the combustion chamber gas.

14   The rate of mass flux and energy flux out of the tank and

15   into the plenum are also modeled utilizing external orifice

16   geometry, tank and plenum temperatures, plenum gas mass and the

17   specific heat of the tank gas. The launch of the vehicle itself

18   is modeled using the mass and energy input into the plenum and

19   subtracting the work imparted upon the vehicle in order to move

20   the vehicle. Finally, losses resulting from blow-by through the

21   gap between the vehicle and the plenum walls, if any, is included

22   in the model.

BRIEF DESCRIPTION OF THE DRAWINGS

1

2   These and other features and advantages of the present

3 invention will become more fully apparent through the following

4 description and appended claims taken in conjunction with the

5 accompanying drawings wherein:

6   FIG. 1 is a schematic of a prior art BAH launcher system;

7   FIG. 2 is a flow chart showing the steps of the method of

8 evaluating a gas generator launcher system in accordance with the

9 teachings of the present invention;

10   FIG. 3 is a conceptual model of a fuel cell burn;

11   FIG. 4 is a block diagram of a computer-based gas generator

12 launcher simulation system;

13   FIG. 5 is a graph of device velocities launched using one

14 and two BAH units according to the system and method of the

15 current invention along with actual test launcher firing data;

16   FIG. 6 is a graph of pressure behind a device launched using

17 one and two BAH units according to the system and method of the

18 current invention along with data obtained from actual test

19 launcher firing;

20   FIG. 7 is a flow chart of an optimization process, which is

21 used to ascertain values for unknown initial conditions; and

1   FIG. 8 is a graph of the results of BAH firings into a 100

2   liter tank, showing results derived through the modeling system

3   and method of the present invention along with experimental

4   values obtained from an actual test launcher firing.

5

6   DESCRIPTION OF THE PREFERRED EMBODIMENT

7   A boosted argon hybrid (BAH) launcher system can be modeled

8   as the combination of fixed volume chambers and an expanding

9   piston chamber connected by sharp edged orifices. FIG. 1, shows

10   a typical BAH launcher system 10. The BAH launcher system 10

11   includes an energy source section 12 and a device launch section

12   14. The energy source section 12 may be removable from the

13   device launch section 14 such that a launcher of the

14   configuration disclosed can be utilized any number of times,

15   provided new energy source sections are attached thereto. In the

16   embodiment shown in FIG. 1, two BAH energy sources 13a and 13b

17   are attached to a single device launch section 14. However, this

18   is not a limitation of the disclosed modeling method, which is

19   flexible enough to model a BAH device launcher configured with

20   any number of BAH energy sources.

21   Each energy source 13, in the preferred embodiment,

22   comprises two chambers, including a combustion chamber 16 and an

1   expansion tank 18. The combustion chamber 16 is in fluid

2   communication with expansion tank 18 via an internal orifice 20.

3   Included in the combustion chamber is a quantity of gas generant

4   fuel 22, which will be more fully described below.

5   The device launch section includes a plenum chamber 24,

6   having interior walls, which is in fluid communication with each

7   expansion tank 18 via an external orifice 26. Also included in

8   the launch section is a device to be launched 28 and an optional

9   ram plate 30, which minimizes losses around the device 28 and

10   between the gap 25 intermediate the device 28 and the walls of

11   the plenum chamber during a launch cycle.

12   In operation, the BAH launcher system 10 has an initial

13   state prior to ignition. In the initial state, each of the

14   combustion chambers 16, expansion tanks 18, and plenum chamber 24

15   have an initial temperature, initial pressure, particular thermal

16   conductivity and a total heat capacity. The internal and

17   external orifices 20, 26, are each capped with a rupture disk,

18   which isolates the three chambers from each other in the initial

19   state. Furthermore, in the initial state, expansion tank 18

20   contains a volume of compressed argon gas. For the purposes of

21   this invention, this gas can be any known relatively inert gas.

22   The combustion chamber 16 has an initial quantity of gas generant

1   22.   The plenum chamber 24 houses a mass to be launched, which

2   includes the mass of vehicle or device 28 and ram plate 30, if

3   included in the system.

4   The gas generant fuel 22 initially has a given total mass

5   whose combustion product mass fractions are known. The fuel also

6   has a known heat capacity, a thermal conductivity, ignition

7   temperature and a flame temperature.

8   The launch process which is modeled by the present invention

9   involves the conversion of the fuel 22 into hot pressurized gas

10   in the combustion chamber 16, the flow of this gas through

11   internal orifice 20 into the tank 18 where compressed argon is

12   contained, the flow of the pressurized argon and gas generated by

13   the fuel through the external orifice 26 into the plenum chamber

14   24, the expansion of the gases in the plenum chamber, the flow of

15   gas through the gap and the device 28 or ram plate 30, and the

16   acceleration of the ram plate 30 and device 28. This basic

17   geometry can be modified to model more complex launcher

18   configurations, without changing the basic physics of the

19   problem.

20   The method of evaluating a gas generator launcher system

21   according to the present invention is shown in FIG. 2. In the

22   first step of the method, step 100, the system geometry and

1   initial conditions are ascertained and fixed. The initial

2   conditions include: a mass, composition and geometry of the fuel

3   included in the gas generant; the geometries of the tank,

4   internal and external orifices and plenum chamber; an initial

5   pressure of the gas within the gas expansion tank; and a mass and

6   geometry of the device to be launched including any gap existing

7   between the device and the walls of the launcher plenum chamber.

8   The second step 110, models a gas generator internal

9   ballistics burn rate and calculates an amount of mass and energy

10   added to the combustion chamber 16 as the fuel is consumed. In

11   step 120 a rate of mass flux out of the combustion chamber 16 is

12   modeled. This model takes into account the internal orifice 20

13   geometry, gas expansion tank and combustion chamber temperatures,

14   tank gas mass, and a ratio of combustion chamber and tank

15   pressures.

16   In step 130, a rate of energy flux out of the combustion

17   chamber 16 and into the gas expansion tank 18 is calculated

18   utilizing the rate of mass flux out of the combustion chamber and

19   into the tank, the combustion chamber temperature and the

20   specific heat of the combustion chamber gas.

21   Next, in step 140, a rate of mass flux out of the gas

22   expansion tank 18 and into the plenum chamber 24 is calculated.

1   This calculation takes into account the external orifice

2   geometry, which separates the gas expansion tank from the plenum,

3   expansion tank and plenum temperatures, plenum gas mass and a

4   ratio of expansion tank and plenum pressures.

5   In step 150, a rate of energy flux out of the expansion tank

6   18 and into the plenum chamber 24 is calculated. This

7   calculation utilizes the rate of mass flux out of the tank and

8   into the plenum as well as the tank temperature and the specific

9   heat of the expansion tank gas.

10   Finally, in step 160, the launch of a vehicle 28 within the

11 plenum chamber 24 is modeled utilizing conservation of mass and

12 energy principles, taking into account the mass and energy input

13 into the plenum and subtracting the losses resulting from blow-by

14 through the gap 25 between the vehicle 28 and the plenum walls.

15   To correctly model the process outlined above, a set of

16 governing equations must be established. To simplify the problem

17 the following assumptions will be made: all gas in the system is

18 argon and can be modeled as an ideal gas; the fuel remains at

19 ambient temperature until it is burned; flow through the orifices

20 is adiabatic and compressible; and flow through the gap can be

21 approximated using Fanno flow equations.

1   The notation used to describe these processes is as follows:

2   The set of equations to be solved are derived using

3   1.   The Gas Generator Internal Ballistics model for the fuel

4   burn rate;

5   2.   Conservation of mass for each of the system volumes;

6   3.   Conservation of energy for each of the system volumes;

7   4.   Compressible flow equations for the flow through orifices

8   and gaps; and

9   5.   Newtonian model for the device motion.

10   Note that the notation used to describe these processes is

11 as follows:

12   All physical quantities related to the combustion chamber

13 will be denoted with the subscript c; all physical quantities

14 related to the tank chamber will be denoted with the subscript 1;

15 all physical quantities related to the plenum chamber will be

16 denoted with the subscript 2; all mass quantities are expressed

17 using m; all pressures are expressed using P; all temperatures

18 are expressed using T; and other variables will be defined as

19 needed.

20   The fuel burn rate is a function of the gas generant fuel

21 slug design and the combustion chamber conditions. As the gas

22 generant fuel burns, the exposed area of the fuel changes. The

1   fuel cell can be thought of as a bar 40 of varying thickness

2   which is burnt from one end to the other. The distance the flame

3   has progressed along the bar length is denoted "s" and the area

4   variation along the bar is expressed as A(s). This conceptual

5   model is illustrated in FIG. 3.

6   Following the Gas Generator Internal Ballistics burn rate

7   model, the rate at which the flame proceeds along the bar is a

8   function of the combustion chamber pressure P_c, i.e.:

9
$$\frac{ds}{dt} = \beta P_c^\alpha \qquad\qquad (1)$$

10   where β is the burn rate coefficient and α is a burn

11   rate exponent that is a constant for the fuel. As the fuel

12   burns, mass is added to the system $\left(\dot{m}_f\right)$ at a rate equal to the

13   consumption rate of the fuel multiplied by the fraction of the

14   combustion products that are gaseous, G. The energy released by

15   the fuel $\left(\dot{E}_f\right)$ is assumed to be proportional to the mass consumed.

16

17
$$\dot{m}_f = \left(\frac{ds}{dt} A(s)\rho_f\right)G \qquad\qquad (2)$$

18

19
$$\dot{E}_f = C_p T \frac{ds}{dt} A(s)\rho_f \qquad\qquad (3)$$

13

1   where C_p is the specific heat of the combustion gasses and   is

2   the combustion chamber temperature and ρ_f is the density of the

3   fuel.   The function A(s) is assumed to be a linear function of s,

4   A(s) = A_0 + A_1 s, for most fuel cell designs.

5   The rate of mass and energy flux out of the combustion

6   chamber is related to the flow rate of gas through the internal

7   orifice.   If the ratio of the tank pressure to the combustion

8   chamber pressure is less than the critical pressure ratio, Pr_{crit},

9   as defined by

10
$$\mathrm{Pr}_{crit} = \left[\frac{2}{\gamma + 1} \right]^{\gamma/(\gamma - 1)} \tag{4}$$

11   then the flow through the orifice will be choked. γ is the ratio

12   of specific heats.   In this case the velocity of the gas through

13   the orifice will be at Mach 1.   The conditions at the orifice

14   throat (for both choked and subsonic flow) are determined using

15   the relations,

16
$$T_t = \frac{T_{upstream}}{\left(1 + \frac{(\gamma - 1)M_t^2}{2} \right)} \tag{5}$$

17
$$P_t = P_{upstream} \left[\frac{1.0}{1 + \frac{(\gamma - 1)}{2} M_t^2} \right]^{\gamma/\gamma - 1} \tag{6}$$

14

1 $$\rho_t = \frac{P_t}{RT_t} \tag{7}$$

2

3 $$C_t = \sqrt{\gamma RT_t} \tag{8}$$

4 where M_t is the Mach number, and

5   R is the gas constant

6 The mass flux out of the combustion chamber can then be expressed

7 as

8 $$\dot{m}_c = A_c \rho_t c_t \tag{9}$$

9 where A_c is the area of the combustion chamber

10 If the pressure ratio is greater than the critical ratio (but

11 less than 1.0), the flow is subsonic. Equations 5 through 9

12 above hold under these conditions, except the Mach number, M_t,

13 must be determined using,

14

15 $$M_t = \left\{ \left[\left(\frac{P_{upstream}}{P_{upstream}} \right)^{(\gamma-1)/\gamma} - 1 \right] \frac{2}{\gamma-1} \right\}^{\frac{1}{2}} \tag{10}$$

16 If the downstream pressure is greater than the upstream pressure,

17 the flow will be reversed and equations 5 through 10 must be

18 adjusted accordingly.

1   The energy flux out of the combustion chamber, $\dot{E}_c$, can be

2   determined using the mass flow rate, the combustion chamber

3   temperature, and the specific heat of the combustion chamber gas,

4   i.e.,

5    $$\dot{E}_c = \dot{m}_c T_c C_p \qquad (11)$$

6   The mass and energy flux into and out of the tank volume can

7   be determined using equations 5 through 10, except instead of

8   considering the combustion chamber and tank pressures and

9   temperatures, the tank and plenum states should be used.

10   The plenum chamber has a mass and energy input from the BAH

11   expansion tank, and it loses mass and energy via work done on the

12   ram plate and device and blow-by through the gap around the ram

13   plate and device. For some configurations, the ram plate gap is

14   very small and the blow-by can be neglected. However in other

15   launcher configurations, there is no ram plate, and there is

16   substantial blow-by around the device. In these configurations

17   the blow-by cannot be neglected. Fanno flow equations are used

18   to compute the mass flow rate through the gap $\left(\dot{m}_{bb}\right)$.

19   Energy lost from gas which escapes around a ram plate, or

20   the device itself in the case where there is no ram plate, is

21   modeled as Fanno flow to account for viscous effects of gas

1 flowing down an annulus. The following describes the case where

2 there is no ram plate and flow passes around the device. The

3 annular area between the device and the tube wall is modeled as

4 an equivalent area nozzle with a length equal to the length of

5 the straight portion of the device. The length of the tapered

6 section of the device, which is relatively short with respect to

7 the overall length of the device, was neglected. The system

8 equations are derived by using the continuity equation at the

9 nozzle throat and the isentropic relations to determine the mass

10 flow rate. The Mach number at the exit of the nozzle is

11 determined to be:

12

13
$$\left(M_2\right)^2 = \left\{\left(\frac{1}{\gamma-1}\right)^2 + \frac{2}{\gamma-1}\frac{M_1^{\ 2}}{\left(\frac{P_{down}}{P_{up}}\right)^2}\left[1+\frac{\gamma-1}{2}M_1^{\ 2}\right]^{-\left(\frac{\gamma+1}{\gamma-1}\right)}\right\}^{\frac{1}{2}} - \frac{1}{\gamma-1} \tag{12}$$

14 Where: $\left(M_2\right)^2$ is the exit Mach number squared

15            $\left(M_1\right)^2$ is the entrance Mach number squared

16   γ is the specific heat ratio of the gas, and

17            $\left(\frac{P_{down}}{P_{up}}\right)$ is the ratio of the downstream pressure to the

18   upstream pressure.

1   Note the use of an upstream and downstream pressure vice

2   using ambient and tube pressure. A distinction is made to allow

3   for the back flow that occurs towards the end of the transient as

4   the plenum pressure drops below ambient, thus reversing the

5   direction of the blow-by flow.

6   Using the Fanno Flow equations, the Mach number at the

7   entrance of the nozzle was determined to be:

8
$$M_1^2 = \left\{ \frac{\gamma 4 f l}{D} + \frac{1}{M_2^2} - \frac{\gamma+1}{2} \ln\left[\frac{M_1^2}{M_2^2} \frac{2+(\gamma-1)M_2^2}{2+(\gamma-1)M_1^2} \right] \right\}^{-1}$$
(13)

9   Where: f is the friction factor;

10   l is the frictional length of the device being

11   launched; and

12   D is the effective diameter and is equal to 4 (wetted

13   area/perimeter).

14   This friction factor is a function of Reynolds number and

15   relative roughness of the walls of the nozzle. Over the range of

16   values expected for flow through the blow-by area between the

17   device and the tube, from Mach number 0.1 to 1, a friction

18   coefficient of between 0.02 and 0.038 is estimated.

19   To solve for the entrance mach number needed in the final

20   mass flow rate calculation, numerical methods were used with the

21   two equations above to iterate to a solution. With the entrance

1 Mach number known, the blow-by mass flow rate can be determined

2 from:

$$\dot{m}_{bb} = \sqrt{\gamma \rho_{up} P_{up}} \, A M_1 \left[1 + \frac{\gamma - 1}{2} M_1^{\,2} \right]^{-\frac{1}{2}\left(\frac{\gamma+1}{\gamma-1}\right)} \tag{14}$$

4 Where: $\dot{m}_{bb}$ is the mass flow rate;

5 ρ_{up} is the upstream gas density; and

6 A is the area of the nozzle or in this case the area

7 of the annulus.

8 The work done by the gas on the ram plate (or device) is given

9 by:

$$\dot{W} = P_2 A_d \dot{x}_2. \tag{15}$$

11 where A_d is the area of the device or ram plate

12 The system dynamics equations can be developed using the

13 mass and energy flux terms developed above. For ease of

14 interpretation, the conservation of energy equations are

15 expressed in terms of the temperature. The internal energy of a

16 volume of gas is given as E = mTC_v. The time derivative of this

17 equation is correctly written as

18

$$\frac{dE}{dt} = C_v \left(m \frac{dT}{dt} + T \frac{dm}{dt} \right) = \dot{E}_{in} \dot{E}_{out} - \dot{W} - \dot{Q} \tag{16}$$

19

1   A heat transfer term $\dot{Q}$ has been included in this expression

2 approximated as HA $(T - T_\infty)$, which includes a heat transfer

3 coefficient H, a representative area A (approximated as the

4 volume to the 2/3rd power), and the temperature difference

5 between the gas and ambient conditions. Although this is a

6 simplification of the heat transfer process, it is a valid

7 approximation of the gross energy loss in the system.

8   Thus, the mass and energy flux terms for the three chambers

9 in the BAH system are:

10 <u>COMBUSTION CHAMBER</u>;

11
$$\frac{dm_c}{dt} = \dot{m}_f - \dot{m}_c \tag{17}$$

12
$$\frac{dT_c}{dt} = \frac{\left(C_p T^* \dot{m}_f - C_p T_c \dot{m}_c - \left(HV_c^{2/3}\right)T\right)_c}{m_c C_v} - \left(\dot{m}_f - \dot{m}_c\right)\left(\frac{T_c}{m_c}\right) \tag{18}$$

13 <u>TANK EXPANSION</u>;

14
$$\frac{dm_t}{dt} = \dot{m}_c - \dot{m}_t \tag{19}$$

15
$$\frac{dT_t}{dt} = \frac{\left(C_p T_c\right)\dot{m}_c - C_p T_t \dot{m}_t - \left(HV_t^{2/3}\right)T_t}{m_t C_v} - \left(\dot{m}_c - \dot{m}_t\right)\left(\frac{T_t}{m_t}\right) \tag{20}$$

16 <u>PLENUM CHAMBER</u>;

17
$$\frac{dm_p}{dt} = \dot{m}_t - \dot{m}_{bb} \tag{21}$$

1 $$\frac{dT_p}{dt} = \frac{\left(C_p T_t \dot{m}_t - C_p T_p m_{bb} - W - \left(\dot{H}V_p^{2/3}\right)T_p\right)}{m_p C_v} - \left(\dot{m}_t - \dot{m}_{bb}\right)\left(\frac{T_p}{m_p}\right) \qquad (22)$$

2   The chamber gas dynamics are coupled to the dynamics of the

3   launch device via the expansion of the plenum chamber. The

4   motion of the launch device is determined using the Newtonian

5   equations of motion, i.e., F=ma. The true mass which must be

6   accelerated is the sum of the device mass (M_d), the added mass of

7   the fluid surrounding the device (M_{add}), and the ram plate and any

8   other parasitic components ($M_{parasitic}$). The forces on the device

9   include the pressure acting on the ram plate, frictional forces,

10   hydrodynamic drag forces, and damping forces. Most launcher

11   configurations can be properly modeled with a combination of

12   these forces. For incorporation into the launcher evaluation

13   model, the following two equations are used:

14

15 $$\ddot{x}_w = \frac{d\dot{x}_2}{dt} = \frac{\left(P_p - P_{amb}\right)A_d - F_{friction} - B\dot{x}_2 - C\dot{x}_2^2}{\left(M_d + M_{add} + M_{parasitic}\right)} \qquad (23)$$

16

17 $$\dot{x}_2 = \frac{dx_2}{dt} \qquad (24)$$

18 where B is a constant for Hydrodynamic drag forces, and C is a

19 damping force constant.

1   At any point during the expansion process, the plenum volume

2   can be computed using the ram plate displacement and the plenum

3   chamber cross-sectional area. For some launchers and launch

4   conditions, the ram plate may separate from the device and a

5   second set of dynamics equations must be developed. The initial

6   set of motion equations must be adjusted to remove the device

7   mass, the fluid added mass, and any external forces imposed

8   directly on the vehicle.

9   A second set of motion equations will govern the dynamics of

10   the vehicle after it separates from the ram plate. After

11   separation, these dynamic equations will not influence the BAH.

12   They will only relate the acceleration of the vehicle due to

13   hydrodynamic and other external forces. This system complication

14   allows the model to estimate the effects of a short power stroke

15   and then a free-flying vehicle.

16   The last dynamics equation that must be included relates to

17   the change of volume in the combustion chamber as the fuel is

18   consumed. The rate of change of the combustion chamber volume

19   is:

20
$$\frac{dV_c}{dt} = \left(\frac{ds}{dt} A(s)\right) G. \tag{25}$$

1   The last set of equations needed to complete the model are

2   the ideal gas state equations for each of the system volumes. It

3   is assumed that the mass, pressure, temperature and volume are

4   related using the equation

5

6                               $$PV = mRT \qquad (26)$$

7   The above-derived system of equations can be solved using a

8   computer system programmed using high level programming

9   languages. The computer-based gas generator launcher simulation

10 system disclosed herein is shown in FIG. 4 and includes: a means

11 for inputting gas generator launcher geometries and initial

12 conditions and a launcher simulator program in computer memory

13 for resolving the above-specified gas and device dynamic

14 equations to integrate a solution from said initial conditions to

15 the end of a launch cycle. The system also includes a display

16 means for displaying the integrated solution. For the present

17 invention, a MATLAB program was developed, which makes use of

18 second and third order Runge-Kutta formulas with variable time

19 steps to integrate the solution from the initial conditions to

20 the end of the launch cycle. In the MATLAB program, two BAH

21 units are allowed, fired with prescribed time delay between them.

1 Consequently, two sets of BAH differential equations are

2 incorporated into the complete model.

3   However, before the system of equations in the preceding

4 section can be solved, the system geometry must be fixed and the

5 initial conditions must be established. Table 1 lists all of the

6 initial conditions that must be established in order to

7 effectively model a specific BAH launcher system.

8

9   Example 1.

10   Table 2 lists the initial conditions utilized in order to

11 evaluate one BAH launcher system utilizing the disclosed

12 evaluation method. Using the initial parameters of Table 2, the

13 launch of a 73 Kg device from a 0.159m diameter tube was

14 simulated for one and two BAH units. An actual BAH launcher

15 using the same initial conditions, was test fired using one and

16 two BAH units. The results of these one and two BAH unit firing

17 simulations are shown in FIG. 5 and 6 along with experimental

18 data for the same conditions. FIG. 5 shows the velocity result

19 for one and two BAH unit firing simulations. Actual data for the

20 two BAH unit design is indicated by square data points such as

21 310. The simulated data for the two BAH unit designs is

22 indicated as line 300. Actual data for the one BAH unit design

24

1   is indicated by triangular data points 330. The simulated data

2   for the one BAH unit design is indicated as line 320.   The

3   agreement can be seen to be very good.

4   Some discrepancies, O(15%), however, are found in the

5   pressures behind the device(s) as shown in FIG. 6 wherein the

6   square data points 350 represent the actual pressure data for the

7   two BAH unit design while the simulated data is represented by

8   line 340, and the triangular data points 370 represent the actual

9   pressure data for the one BAH unit design while the simulated

10 pressure data for a one BAH unit is represented by line 360.

11   Additional factors can be added to the model to account for

12 these discrepancies. The heat transfer model can be modified to

13 account for transient effects in the system structure, the

14 transport and thermal capacitance of solid particles variation in

15 the orifice geometry from BAH to BAH among others. In addition,

16 there is some uncertainty in the experimental data which has not

17 been quantified.

18   Nonetheless, the disclosed evaluation methodology allows BAH

19 launcher designers the ability to evaluate launcher designs and

20 predict, with sufficient accuracy, the performance

21 characteristics of their designs. Thus, unacceptable designs can

1   be ruled out early on in the design process and only those

2   designs for which acceptable performance characteristics can be

3   predicted can be pursued further in the design process.

4   Even when all launcher initial parameters are not known

5   explicitly, the disclosed launcher evaluation methodology can aid

6   designers in optimizing values for the unknown parameters. For

7   example, while launcher and device geometries will be explicitly

8   known, certain parameters related to the BAH units themselves may

9   not be available to designers for any number of reasons. (Of the

10   values indicated in Table 2, parameters 27 through 36, which

11   **TABLE 1**

12   Initial Conditions for BAH Launcher System

13

VARIABLE	DESCRIPTION
ρ_s	Density of solid fuel
A_d	Device frontal area
D_p	Ram plate diameter
M_{add}	Fluid added mass
$M_{parasitic}$	Parasitic mass
P_{po}	Initial plenum pressure
T_{po}	Initial plenum temperature
T_{to}	Initial BAH internal temperature
V_t	BAH tank volume
X_{20}	Initial ram plate displacement
M_{po}	Initial mass of plenum gas
M_{co}	Mass of gas in combustion chamber
M_{to}	Mass of gas in tank
γ	C_p/C_v
A_n	Flow loss area (launcher specific)
B	Device damping coefficient
C_{dyn}	Hydrohynamic drag coefficient of device
C_n	Flow loss coefficient (launcher specific)
C_p	Specific heat of gas, fixed pressure

C_v	Specific heat of gas, fixed volume
f	Device friction coefficient
M_d	Device mass
M_f	Mass of combustible fuel
M_g	Molecular weight of gas
R	Gas constant
V_{po}	Initial plenum volume
α	Burn rate exponent
β	Burn rate coefficient
A0	Solid fuel shape coefficient, offset
A1	Solid fuel shape coefficient, slope
A_c	Orifice area at combustion chamber exit
A_t	Orifice area at tank exit
P_{to}	Initial BAH internal pressures
T	Enthalpy of combustion/Cp
H	Internal heat transfer coefficient
V_c	Combustion chamber volume

1

2

3

TABLE 2

4   Simulation Parameters for BAH Launcher System of Example 1

5

No.	VARIABLE	DESCRIPTION	VALUE
1	ρ_s	Density of solid fuel	1681 kg/m^3
2	A_d	Device frontal area	1.986e-2 m^2
3	D_p	Ram plate diameter	0.159m
4	M_{add}	Fluid added mass	0 kg
5	$M_{parasitic}$	Parasitic mass	0 kg
6	P_{p0}	Initial plenum pressure	1.013e5 Pa
7	T_{p0}	Initial plenum temperature	273.2 K
8	T_{t0}	Initial BAH internal temperature	273.2K

9	V_t	BAH tank volume	4.81e-4 m³
10	X_{20}	Initial ram plate displacement	0
11	M_{p0}	Initial mass of plenum gas	Eqn 26
12	M_{c0}	Mass of gas in combustion chamber	Eqn 26
13	m_{t0}	Mass of gas in tank	Eqn 26
14	γ	C_p/C_v	1.561
15	A_n	Flow loss area	0
16	B	Device damping coefficient	0
17	C_{dyn}	Hydrohynamic drag coeff. of device	0
18	C_n	Flow loss coefficient	0
19	C_p	Specific heat of gas, fixed press.	487.3 J/kg/k
20	C_v	Specific heat of gas, fixed vol.	312.2 J/kg/K
21	f	Device friction coefficient	0
22	M_d	Device mass	73 kg
23	M_f	Mass of combustible fuel	0.026 kg
24	M_g	Molecular weight of gas	39.24kg/kmole
25	R	Gas constant	211.9 J/kg/K
26	V_{p0}	Initial plenum volume	0.0227 m³
27	α	Burn rate exponent	0.2577
28	β	Burn rate coefficient	0.5224
29	A0	Solid fuel shape coeff., offset	0.29m²

30	A1	Solid fuel shape coeff., slope	1.664m
31	A_c	Orifice area: combustion chamber exit	2.14e-5 m^2
32	A_t	Orifice area: tank exit	7.00e-5 m^2
33	P_{t0}	Initial BAH internal pressures	2.55e7 Pa
34	T	Enthalpy of combustion/Cp	3443 K
35	H	Internal heat transfer coefficient	846.7 $W/K/m^2$
36	V_c	Combustion chamber volume	1.37e-4 m^3

1

2 relate specifically to the BAH unit(s) themselves, were initially

3 unknown.) To determine these values, a simplified model was developed

4 for the discharge of a BAH into a tank by adjusting the known launch

5 system parameters as necessary and an optimization process, as shown

6 in FIG. 7, was performed to determine the "best estimates" for the

7 unknown variables. The optimization process 200 begins with entering

8 known initial conditions, step 210 and entering reasonable hypotheses

9 for the unknown variables, step 220. Equations 1 through 23 are then

10 solved using the MATLAB program, step 230. Upon completion of the

11 simulation, the results are compared to experimental data obtained

12 from test firings of a BAH unit in a similar configuration, step 240.

13 The root mean squared (RMS) error of the simulation is then recorded,

14 step 250. The results of the current simulation are compared to the

1   results of previous simulations, step 260. If the variables used in

2   the most current simulation produce a result that more accurately

3   matches the experimental data than the preceding simulation, then the

4   new variables are recorded as the current "best estimate", step 270.

5   The unknown variables are then randomly changed, step 280.

6   The simulation is then repeated with the new variables and

7   the accuracy of the simulation is compared to the preceding

8   simulation. This process is then repeated a large number of

9   times until the solution converges at a point of minimum error.

10 Although this "random walk" optimization process is functional,

11 its accuracy depends, in large part, on the range of variables

12 allowed and the number of times the equations are solved using

13 the different variables. Nonetheless, once the unknown

14 parameters are optimized, their "best estimate" values can be

15 utilized in the disclosed evaluation methodology for any number

16 of additional launcher configurations that use substantially the

17 same BAH units.

18   FIG. 8 plots a comparison of the simulated BAH firing using

19 the disclosed evaluation methodology, curve 380, and experimental

20 data, points 390 recorded during an actual BAH launch cycle. The

21 pressures in a 100 liter tank are shown versus time. As can be

22 seen, the disclosed evaluation methodology predicts the actual

1   data extremely well (error of the order of 1%). The experimental

2   data has been extrapolated in time to improve the prediction

3   accuracy near the end of the combustion simulation.

4   In light of the above, it is therefore understood that

5   the invention may be

6   practiced otherwise than as specifically described.

2

3   COMPUTER IMPLEMENTED SYSTEM AND METHOD

4   FOR EVALUATING GAS GENERATOR LAUNCHERS

5

6   ABSTRACT OF THE DISCLOSURE

7   A computer-implemented system and method for evaluating gas

8 generator launchers is provided. The method includes fixing the

9 system geometry and establishing initial conditions of the

10 launcher to be evaluated. The initial conditions include a mass,

11 composition and geometry of the fuel included in the gas

12 generant, geometries of the system components, initial pressures

13 and temperatures and the mass and geometry of the device to be

14 launched. A gas generator internal ballistics burn rate is

15 modeled and an amount of mass and energy added to the combustion

16 chamber as the fuel is consumed is calculated. Then, using

17 conservation of mass and energy principles, an energy flux rate

18 is modeled, beginning with the fuel and ending with the work

19 performed on the device in order to propel it from the launcher.

20 The modeling method is performed using a computer-based gas

21 generator launcher simulation system, which includes a means for

22 inputting gas generator launcher geometries and initial

1 conditions, a launcher simulator program in computer memory for

2 resolving gas and device dynamic equations to integrate a

3 solution from said initial conditions to the end of a launch

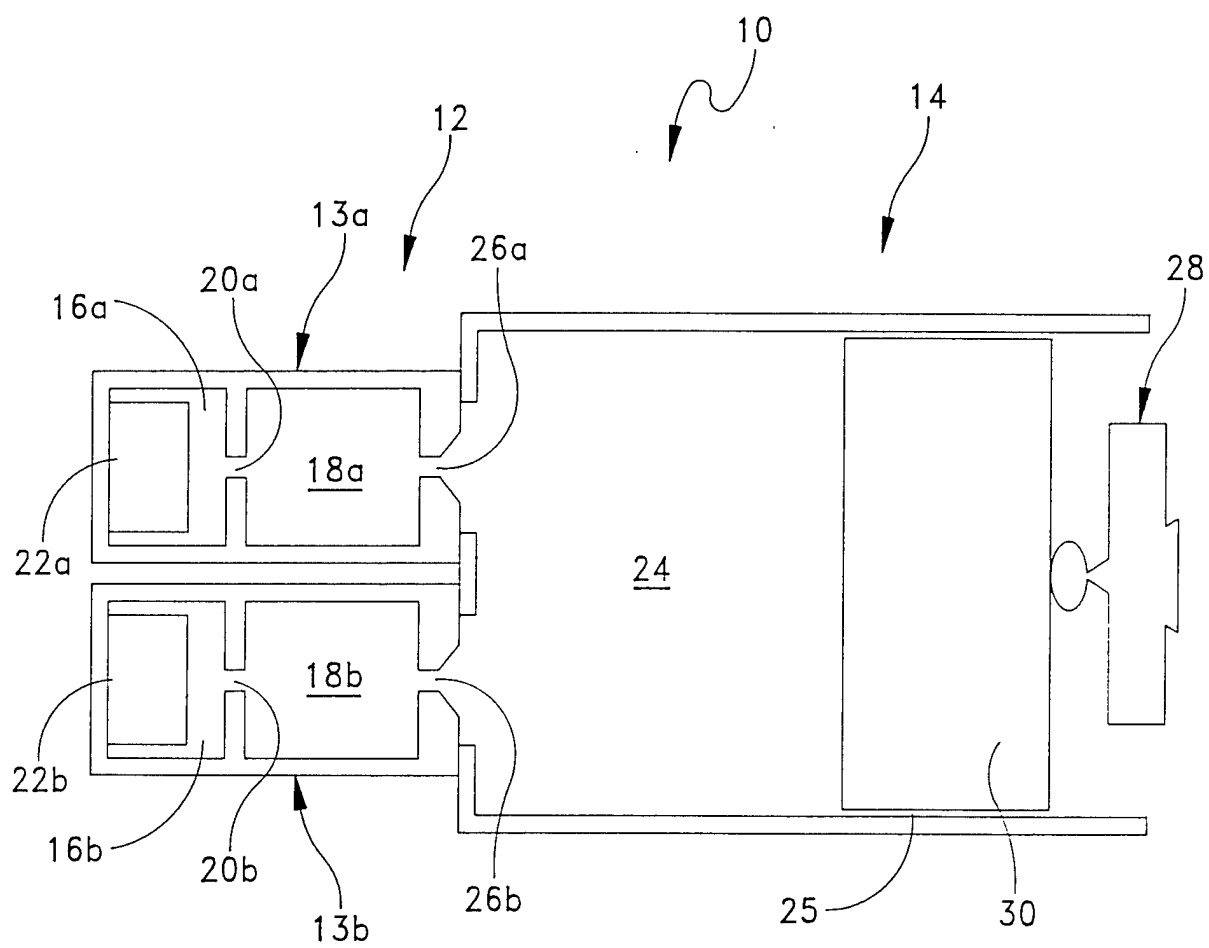4 cycle, and a display means for displaying the integrated
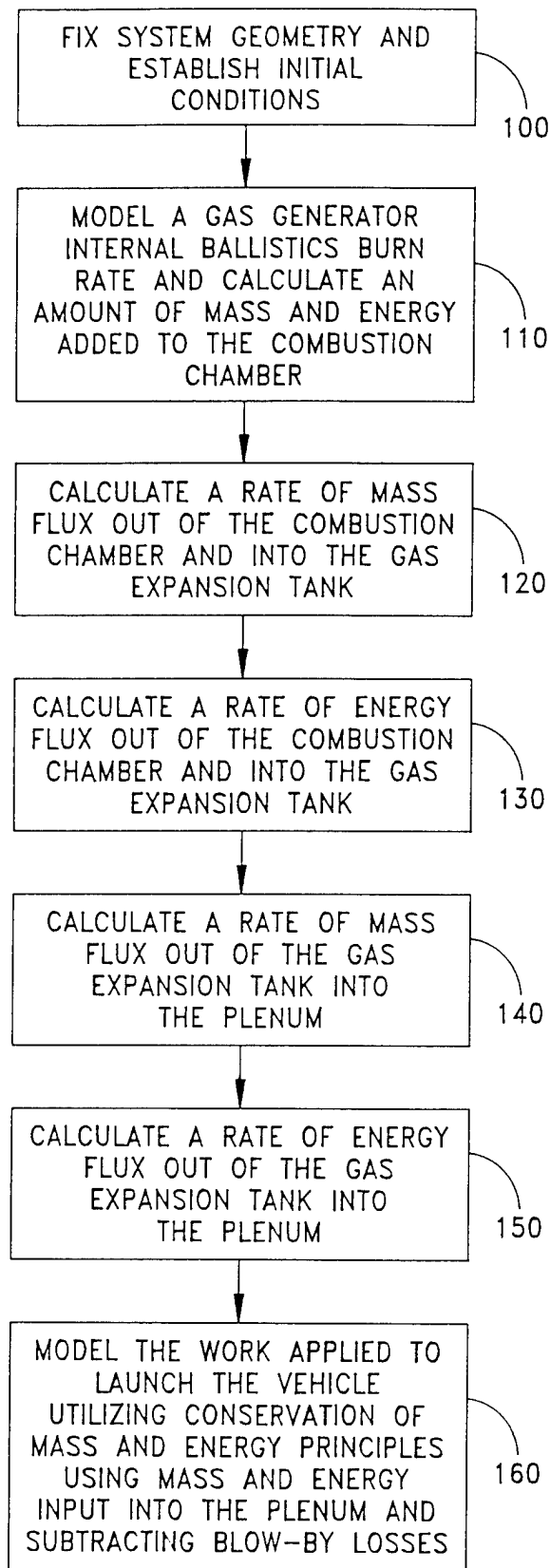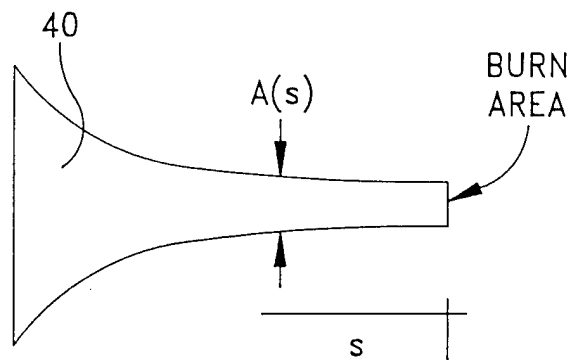
5 solution.

FIG. 1

```
┌─────────────────────────┐
│   FIX SYSTEM GEOMETRY AND │
│      ESTABLISH INITIAL    │───╮
│        CONDITIONS         │   │  100
└─────────────────────────┘   ╯
            │
            ▼
┌─────────────────────────┐
│    MODEL A GAS GENERATOR  │
│  INTERNAL BALLISTICS BURN │
│    RATE AND CALCULATE AN  │───╮
│  AMOUNT OF MASS AND ENERGY│   │  110
│   ADDED TO THE COMBUSTION │   ╯
│          CHAMBER          │
└─────────────────────────┘
            │
            ▼
┌─────────────────────────┐
│   CALCULATE A RATE OF MASS│
│  FLUX OUT OF THE COMBUSTION│──╮
│  CHAMBER AND INTO THE GAS │   │  120
│      EXPANSION TANK       │   ╯
└─────────────────────────┘
            │
            ▼
┌─────────────────────────┐
│  CALCULATE A RATE OF ENERGY│
│  FLUX OUT OF THE COMBUSTION│──╮
│   CHAMBER AND INTO THE GAS│   │  130
│       EXPANSION TANK      │   ╯
└─────────────────────────┘
            │
            ▼
┌─────────────────────────┐
│   CALCULATE A RATE OF MASS│
│      FLUX OUT OF THE GAS  │───╮
│     EXPANSION TANK INTO   │   │  140
│         THE PLENUM        │   ╯
└─────────────────────────┘
            │
            ▼
┌─────────────────────────┐
│  CALCULATE A RATE OF ENERGY│
│      FLUX OUT OF THE GAS  │───╮
│     EXPANSION TANK INTO   │   │  150
│         THE PLENUM        │   ╯
└─────────────────────────┘
            │
            ▼
┌─────────────────────────┐
│   MODEL THE WORK APPLIED TO│
│      LAUNCH THE VEHICLE   │
│  UTILIZING CONSERVATION OF │───╮
│ MASS AND ENERGY PRINCIPLES │   │  160
│   USING MASS AND ENERGY   │   ╯
│  INPUT INTO THE PLENUM AND│
│ SUBTRACTING BLOW-BY LOSSES│
└─────────────────────────┘
```

FIG. 2

40

A(s)

BURN
AREA

s

FIG. 3

MEANS FOR INPUTTING
GAS GENERATOR
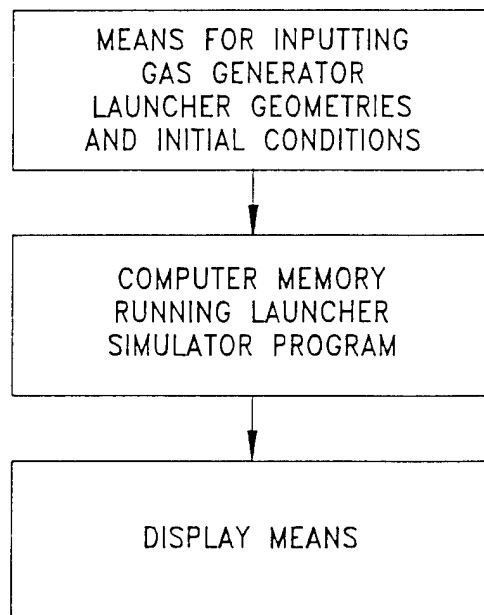LAUNCHER GEOMETRIES
AND INITIAL CONDITIONS

COMPUTER MEMORY
RUNNING LAUNCHER
SIMULATOR PROGRAM

DISPLAY MEANS

FIG. 4

FIG. 5

FIG. 6

```
ENTER KNOWN INITIAL
CONDITIONS                    210

         ↓

ENTER A REASONABLE
HYPOTHESIS FOR EACH
UNKNOWN INITIAL
CONDITION                     220

         ↓

SOLVE EQUATIONS 1-23
ACCORDING TO STEPS 110
THROUGH 160 OF FIGURE 2       230

         ↓

COMPARE RESULTS WITH
EXPERIMENTAL DATA
OBTAINED FROM TEST
FIRINGS                       240

         ↓

RECORD RMS ERROR OF
CURRENT SIMULATION            250

         ↓

COMPARE RESULTS OF
CURRENT SIMULATION
WITH RESULTS OF
PREVIOUS SIMULATION           260

         ↓

IF CURRENT SIMULATION
MATCHES TEST FIRING
BETTER THAN PREVIOUS,
SAVE VALUES USED FOR
UNKNOWNS IN CURRENT
SIMULATION AS "BEST
ESTIMATE" VALUES              270

         ↓

RANDOMLY CHANGE UNKNOWN
VARIABLES                     280
```

200

FIG. 7

FIG. 8