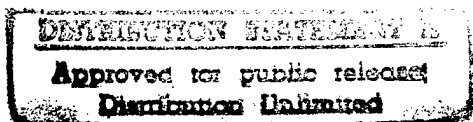


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Inventor                 Robert A. Benson, Jr.

NOTICE

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DEPARTMENT OF THE NAVY  
CODE OCCC  
ARLINGTON VA 22217-5660



DTC QUALITY INSPECTED 3

19980105 086

1 Navy Case No. 77994

2

3 UNDERSEA VEHICLE PROPULSION AND ATTITUDE CONTROL SYSTEM

4

5 STATEMENT OF GOVERNMENT INTEREST

6 The invention described herein may be manufactured and  
7 used by or for the Government of the United States of  
8 America for governmental purposes without the payment of any  
9 royalties thereon or therefore.

10

11 BACKGROUND OF THE INVENTION

12 (1) Field Of The Invention

13 The present invention relates to vehicle propulsion  
14 systems and more particularly, a system for controlling both  
15 propulsion and attitude or orientation of an undersea  
16 vehicle.

17 (2) Description Of The Prior Art

18 Undersea vehicles are commonly used in the ocean and  
19 other underwater environments for exploration, warfare, and  
20 other purposes. The movement and orientation of these  
21 undersea vehicles, particularly unmanned undersea vehicles,  
22 must be precisely controlled. Unlike surface vessels which  
23 generally move within a single plane on the surface of the  
24 water, undersea vehicles must be capable of moving in  
25 multiple planes and require a system that controls movement  
26 in more "degrees-of-freedom" than that used on surface

1 vessels. In addition to lateral movement, undersea vehicles  
2 have a component of movement in the vertical direction.

3 Typical undersea vehicles are operated at various  
4 speeds in various directions (e.g. lateral, vertical,  
5 forward and reverse) by controlling the propulsion of the  
6 vehicle in those directions. An undersea vehicle must also  
7 be capable of changing directions by controlling the  
8 attitude or orientation of the undersea vehicle, for  
9 example, the pivoting of the vehicle up or down within a  
10 vertical plane (known as "pitch") and the pivoting of the  
11 vehicle from side to side within a horizontal plane (known  
12 as "yaw").

13 To accomplish the additional movement, prior art  
14 undersea vehicles have used numerous separate motors and  
15 propulsors or propulsion devices. For example, controlling  
16 the propulsion and attitude of the vehicle is typically  
17 achieved through the use of forward and aft thruster pairs  
18 and a propulsion motor/propulsor combination. A total of  
19 five separate motors and propulsors are often used to  
20 control the lateral, vertical, forward and reverse motion of  
21 conventional undersea vehicles. Such a large number of  
22 electrical motors occupies a considerable volume of the  
23 undersea vehicle and generates an undesirable amount of  
24 noise.



1           The forward port and aft port preferably extend  
2 generally along a longitudinal axis of the undersea vehicle.  
3 Fluid discharged through the aft port and forward port cause  
4 forward and reverse motion, respectively, of the undersea  
5 vehicle in a direction generally along the longitudinal axis  
6 of the undersea vehicle. Each of the plurality of radial  
7 outlet ports extend along radial lines generally orthogonal  
8 to the longitudinal axis of the undersea vehicle. Fluid  
9 discharged from the plurality of radial outlet ports causes  
10 movement of the undersea vehicle in a radial direction  
11 generally orthogonal to the longitudinal axis of the  
12 undersea vehicle.

13           In one embodiment, the pump is a reversible pump having  
14 a first inlet/outlet connected to the aft port conduit and a  
15 second inlet/outlet connected to the forward port conduit.  
16 According to this embodiment, the plurality of valves  
17 include a first valve connected to the aft port conduit, for  
18 controlling fluid flow between the aft port and the first  
19 inlet/outlet of the pump; a second valve connected between  
20 the aft port conduit and the forward port conduit, for  
21 controlling fluid flow between the second inlet/outlet and  
22 the aft and forward port conduits; a third valve connected  
23 between the radial outlet port conduit and the first  
24 inlet/outlet of the pump, for controlling fluid flow between  
25 the plurality of radial outlet ports and the first  
26 inlet/outlet of the pump; and a fourth valve connected  
27 between the radial outlet port conduit and the second

1 inlet/outlet of the pump, for controlling fluid flow between  
2 the plurality of radial outlet ports and the second  
3 inlet/outlet of the pump.

4 In another embodiment, the pump is a unidirectional  
5 pump having an inlet connected to the forward port conduit  
6 and an outlet connected to the aft port conduit. In this  
7 embodiment, the plurality of valves include: a first valve  
8 connected to the aft port conduit, for controlling fluid  
9 flow between the aft port and the outlet of the pump; a  
10 second valve connected between the aft port conduit and the  
11 forward port conduit, for controlling fluid flow between the  
12 inlet of the pump and the aft and forward port conduits; a  
13 third valve connected between the forward port conduit and  
14 the inlet of the pump, for controlling fluid flow between  
15 the forward port conduit and the pump; and a fourth valve  
16 connected between the radial outlet port conduit and the  
17 forward port conduit, for controlling fluid flow between the  
18 radial outlet port conduit and the forward port conduit.

19 The plurality of radial outlet ports preferably include  
20 forward radial outlet ports disposed proximate the forward  
21 end of the undersea vehicle and aft radial outlet ports  
22 disposed proximate the aft end of the undersea vehicle. The  
23 plurality of radial outlet ports also include at least a  
24 first pair of radial outlet ports disposed on opposite sides  
25 of the undersea vehicle along a first radial line and at  
26 least a second pair of radial outlet ports disposed on

1 opposite sides of the undersea vehicle along a second radial  
2 line generally orthogonal to said first radial line.

3  
4 BRIEF DESCRIPTION OF THE DRAWINGS

5 These and other features and advantages of the present  
6 invention will be better understood in view of the following  
7 description of the invention taken together with the  
8 drawings wherein like reference numerals refer to like parts  
9 and wherein:

10 FIG. 1 is a side schematic view of an undersea vehicle  
11 propulsion and attitude control system according to one  
12 embodiment of the present invention;

13 FIG. 2 is a cross-sectional schematic view of the  
14 undersea vehicle propulsion and attitude control system  
15 taken along line 2-2 in FIG. 1;

16 FIG. 3A is a side schematic view of the undersea  
17 vehicle propulsion and attitude control system according to  
18 the first embodiment of the present invention, for  
19 controlling forward motion of the undersea vehicle;

20 FIG. 3B is a side schematic view of the undersea  
21 vehicle propulsion and attitude control system according to  
22 the first embodiment of the present invention, for  
23 controlling reverse motion of the undersea vehicle;

24 FIG. 3C is a side schematic view of the undersea  
25 vehicle propulsion and attitude control system according to  
26 the first embodiment of the present invention, for  
27 controlling hovering motion of the undersea vehicle;

1           FIG. 4 is a side schematic view of the undersea vehicle  
2 propulsion and attitude control system according to a second  
3 embodiment of the present invention;

4           FIG. 5A is a side schematic view of the undersea  
5 vehicle propulsion and attitude control system according to  
6 the second embodiment for controlling forward motion of the  
7 undersea vehicle;

8           FIG. 5B is a side schematic view of the undersea  
9 vehicle propulsion and attitude control system according to  
10 the second embodiment for controlling reverse motion of the  
11 undersea vehicle; and

12           FIG. 5C is a side schematic view of the undersea  
13 vehicle propulsion and attitude control system according to  
14 the second embodiment for controlling hovering motion of the  
15 undersea vehicle.

16

17                                   DESCRIPTION OF THE PREFERRED EMBODIMENT

18           An undersea vehicle propulsion and attitude control  
19 system 10, FIG. 1, according to the present invention, is  
20 used in an undersea vehicle 12, such as, but not limited to,  
21 an unmanned undersea vehicle, to control the motion of the  
22 undersea vehicle 12 in multiple planes or "degrees of  
23 freedom". The undersea vehicle 12, such as a torpedo or  
24 other unmanned undersea vehicle, preferably includes a  
25 generally cylindrical body having an aft end 14, a forward  
26 end 16 and a longitudinal axis 18. The present invention  
27 contemplates using the propulsion and attitude control



1 system on other types of undersea vehicles having various  
2 shapes.

3 The propulsion and attitude control system 10 includes  
4 a fluid medium pumping device referred to herein as pump 20,  
5 such as a motor/pump jet, disposed in the undersea vehicle  
6 12 that receives and discharges a fluid medium, such as sea  
7 water. Typically, the pump 20 is a water pump, such as the  
8 type used in recreational jet skis, that is driven by an  
9 electric motor and produces about 300 - 600 lbs. of thrust.

10 In the first embodiment, the pump 20 is reversible and  
11 includes at least a first inlet/outlet 22 and at least a  
12 second inlet/outlet 24, both of which take in the fluid  
13 medium or discharge the fluid medium depending upon the  
14 direction in which the pump is operating. An example of a  
15 reversible motor/pump combination is disclosed further in  
16 U.S. Patent No. 5,607,329 to Cho et al. and U.S. Patent  
17 Application Serial No. 08/649,971 (Attorney Docket No. N.C.  
18 77314) filed on May 1, 1996, entitled A Marine Propulsion  
19 System for Underwater Vehicles, and incorporated herein by  
20 reference.

21 The propulsion and attitude control system 10 further  
22 includes at least one forward port 30, at least one aft port  
23 34, and a plurality of radial outlet ports 40, 42 fluidly  
24 coupled to the pump 20. A plurality of valves 50-56 control  
25 fluid flow from the pump 20 through the respective ports.  
26 By opening and closing selected valves 50-56, the propulsion  
27 and orientation of the undersea vehicle 12 is controlled in

1 the forward, reverse, lateral and vertical directions, as  
2 will be described in greater detail below.

3 The forward port 30 is disposed at the forward end 16  
4 of the undersea vehicle 12, preferably but not necessarily  
5 along the longitudinal axis 18. A forward port conduit 32  
6 fluidly connects the forward port 30 to the second  
7 inlet/outlet 24 of the pump 20. The forward port 30 acts as  
8 an inlet for the fluid medium when the undersea vehicle 12  
9 moves in a forward direction indicated by arrow 60 along the  
10 longitudinal axis 18, and as an outlet when the undersea  
11 vehicle 12 moves in a reverse direction indicated by arrow  
12 62, as described in greater detail below.

13 The aft port 34 is disposed in the aft end 14 of the  
14 undersea vehicle 12, preferably along the longitudinal axis  
15 18. An aft port conduit 36 fluidly connects the aft port 34  
16 to the first inlet/outlet 22 of the pump 20. The aft port  
17 34 also acts as either an inlet or outlet for the fluid  
18 medium depending upon the desired motion of the undersea  
19 vehicle 12.

20 The plurality of radial outlet ports 40, 42 are  
21 disposed radially in the undersea vehicle 12 between the  
22 forward end 16 and the aft end 14, preferably along radial  
23 lines 19. A radial outlet port conduit 44 fluidly connects  
24 the plurality of radial outlet ports 40, 42 to the pump 20.  
25 The radial outlet ports 40, 42 discharge fluid to move the  
26 undersea vehicle 12 along respective radial lines 19  
27 generally orthogonal to the longitudinal axis 18.

1           In the embodiment having a reversible pump 20, a first  
2 valve 50 is connected to the aft port conduit 36 for  
3 controlling fluid flow between the aft port 34 and the first  
4 inlet/outlet 22 of the pump 20. A second valve 52 is  
5 connected between the forward port conduit 32 and the aft  
6 port conduit 36, for example, through an intermediate  
7 conduit 38, for controlling fluid flow between the forward  
8 port 30 and the aft port 34. A third valve 54 is connected  
9 between the radial outlet port conduit 44 and the first  
10 inlet/outlet 22 of the pump 20, for controlling flow of the  
11 fluid from the pump 20 into the radial outlet port conduit  
12 44 when the first inlet/outlet 22 of the pump 20 is acting  
13 as an outlet that discharges the fluid.

14           A fourth valve 56 is connected between the radial  
15 outlet port conduit 44 and the second inlet/outlet 24 of the  
16 pump 20, for controlling fluid flow between the pump 20 and  
17 the radial outlet port conduit 44 when the second  
18 inlet/outlet 24 is acting as an outlet that discharges the  
19 fluid (i.e., when the pump 20 is operated in reverse). One  
20 example of the valves includes electrically operated  
21 solenoid valves. The valves can be opened and closed  
22 simultaneously and are preferably timed so that they are  
23 open/closed when the pump is stopped or running at a slow  
24 speed.

25           By selectively opening and closing the valves 50-56 and  
26 controlling the fluid flow out of the forward port 30, aft  
27 port 34, and radial outlet ports 40, 42, the undersea

1 vehicle 12 can be moved in multiple planes of movement or  
2 "degrees of freedom", as described in greater detail below.  
3 The present invention contemplates other combinations or  
4 arrangements of valves that provide an equivalent flow of  
5 fluid medium from the pump 20 to one or more of forward port  
6 30, aft port 34, and radial ports 40, 42.

7 One or more radial outlet port control valves 46, 48  
8 can be coupled to each radial outlet port 40, 42 for  
9 selectively varying the fluid flow or discharge through each  
10 individual radial outlet port 40, 42, for example, by  
11 varying the port orifice to act like tunnel thrusters. The  
12 propulsion and attitude control system 10 preferably  
13 includes a controller 49, such as a standard vehicle linear  
14 controller or a non-linear sliding mode controller as is  
15 well known in the art, for selectively controlling the  
16 valves 50-56 and the radial outlet port control valves 46,  
17 48 and thereby independently controlling the fluid medium  
18 discharge through the forward port 30, aft port 34 and each  
19 of the radial outlet ports 40, 42. One example of the  
20 sliding mode controller includes control software that runs  
21 on the vehicle control computer, such as a Unix operating  
22 system. Independent control of the radial outlet ports 40,  
23 42 thereby controls the pitch and yaw, hover, and  
24 ascent/descent of the undersea vehicle 12, while control of  
25 the forward port 30 and aft port 34 controls motion along  
26 longitudinal axis 18.

1           In the preferred embodiment, the plurality of radial  
2 ports 40, 42 preferably include aft radial ports 40 disposed  
3 proximate the aft end 14 of the undersea vehicle 12 and  
4 forward radial ports 42 disposed proximate the forward end  
5 16 of the undersea vehicle 12. The sets of radial ports 40,  
6 42 are preferably located at a sufficient distance apart to  
7 effectively control the vehicle's pitch and yaw. Each  
8 plurality of radial outlet ports 40, FIG. 2, further  
9 includes a first pair of radial outlet ports 40a, 40b  
10 disposed on opposite sides of the undersea vehicle 12 along  
11 a first radial line 19a, and a second pair of radial outlet  
12 ports 40c, 40d disposed on opposite sides of the undersea  
13 vehicle 12 along a second radial line 19b that is generally  
14 orthogonal to the first radial line 19a. The undersea  
15 vehicle 12 is typical of such vehicles in that the center of  
16 buoyancy  $C_b$  and the center of gravity  $C_g$  are spaced a  
17 distance apart along the radial line 19 on opposite sides of  
18 the longitudinal axis 18. Such a configuration tends to  
19 maintain the radial line 19 oriented in a vertical  
20 direction. In the preferred embodiment of FIG. 2, radial  
21 lines 19a, 19b are rotated  $45^\circ$  from radial line 19.

22           Each of the radial outlet ports 40a-40d can include a  
23 respective control valve 46a-46d, for selectively  
24 controlling the discharge of the fluid and the movement of  
25 the undersea vehicle 12 in numerous planes of movement.  
26 This allows radial ports 40a-d to control pitch, yaw and  
27 roll of the undersea vehicle 12. The effect of discharging

1 fluid from radial ports 40a, 40d is to move the undersea  
2 vehicle 12 vertically downward, while discharging from  
3 radial ports 40b and 40c moves the undersea vehicle 12  
4 vertically upward. Similarly, discharging from pairs of  
5 radial ports 40a,c or 40b,d moves the undersea vehicle 12  
6 laterally to the right or left, respectively. To control  
7 roll, fluid is discharged from pairs of radial ports 40a,b  
8 or 40c,d. The horizontal and vertical components of the  
9 discharges cancel such that the undersea vehicle 12 does not  
10 move vertically or laterally. However, due to the offset  $C_b$   
11 and  $C_g$ , the discharges cause unbalanced moments which rotate  
12 the undersea vehicle 12. Forward radial ports 42 are  
13 configured in a like manner. The radial outlet ports 40, 42  
14 and the associated radial outlet port control valves 46, 48  
15 preferably control the undersea vehicle movement, such as  
16 the pitch, roll and yaw, at slower speeds. Additional  
17 control surfaces/elements, such as rudders and elevators,  
18 can be disposed on the surface of the undersea vehicle 12 to  
19 further control or aid in the control of the pitch, yaw and  
20 roll of the undersea vehicle 12 at higher speeds. It will  
21 be understood that the placement of radial ports 40a-40d can  
22 be configured to suit the characteristics of the particular  
23 undersea vehicle 12 being used. For example, ports 40a-40d  
24 may be located along radial line 19 and along radial line  
25 19c orthongonal to radial line 19.

26 To operate the present propulsion and attitude control  
27 system 10, the plurality of valves 50-56, FIGS. 3A-3C, are

1 selectively opened and closed (opened valves are shown as  
2 white and closed valves are shown as black). The valves 50-  
3 56 are preferably opened/closed by the vehicle controller  
4 49, e.g., a computerized unit with navigation and attitude  
5 control software, as described above with respect to the  
6 radial port control valves 46, 48.

7 To cause forward motion of the undersea vehicle 12,  
8 FIG. 3A, generally in the direction of arrow 60, the first  
9 valve 50 is opened, the second valve 52 is closed, the third  
10 valve 54 is opened, and the fourth valve 56 is closed. The  
11 forward port 30 acts as an inlet that receives the fluid  
12 medium into the forward port conduit 32. The pump 20  
13 receives the fluid medium from the forward port conduit 32  
14 and discharges the fluid medium through the aft port conduit  
15 36 and open first valve 50. The fluid is then discharged  
16 from the aft port 34, creating a rear thrust that moves the  
17 vehicle 12 in a forward direction. The open third valve 54  
18 allows the fluid discharged from the outlet 22 of the pump  
19 20 to flow through the radial outlet port conduit 44,  
20 thereby allowing the fluid to be discharged through one or  
21 more of the radial outlet ports 40, 42, as necessary, to  
22 control the direction and orientation of the undersea  
23 vehicle 12.

24 To provide a reverse motion to the undersea vehicle 12,  
25 FIG. 3B, generally in the direction of arrow 62, the  
26 operation of pump 20 is reversed and the aft port 34 acts as  
27 an inlet that receives the fluid medium. The first valve 50

1 is opened and the second valve 52 is closed so that the  
2 fluid received in the aft port 34 is transferred through the  
3 aft port conduit 36 to the pump 20 which discharges the  
4 fluid medium to the forward port conduit 32 and out of the  
5 forward port 30, acting as the outlet. The third valve 54  
6 is closed and the fourth valve 56 is opened so that a  
7 portion of the fluid medium discharged from the outlet 24 of  
8 the reversed pump 20 is directed to the radial outlet port  
9 conduit 44.

10 To provide a hover motion to the undersea vehicle 12,  
11 FIG. 3C, generally in the directions of arrows 64, 66, the  
12 first valve 50 is closed, the second valve 52 is opened, the  
13 third valve 54 is opened, and the fourth valve 56 is closed.  
14 By closing the first valve 50 and opening the second valve  
15 52 between the aft port conduit 36 and forward port conduit  
16 32, both the aft port 34 and forward port 30 act as inlets  
17 and the forward and reverse motion is nulled by the pump 20.  
18 Opening the third valve 54 allows the fluid medium  
19 discharged from the pump 20 from aft and forward port inlets  
20 34, 30, to be directed to the radial outlet port conduit 44,  
21 thereby providing nulling movement of the undersea vehicle  
22 12. As discussed above, individual control of the radial  
23 outlet ports 40, 42 allows the undersea vehicle 12 to be  
24 moved upwardly, downwardly, or laterally to various depths  
25 or locations within an undersea environment.

26 In a second embodiment of the undersea vehicle  
27 propulsion and attitude control system 110, FIG. 4, the pump



1 120 is unidirectional and includes an inlet 124 for  
2 receiving the fluid medium and an outlet 122 for discharging  
3 the fluid medium. Similar to the first embodiment, a  
4 forward port conduit 132 fluidly connects the inlet 124 to a  
5 forward port 130. An aft port conduit 136 fluidly connects  
6 the outlet 122 to an aft port 134. A radial outlet port  
7 conduit 144 fluidly connects the radial outlet ports 140,  
8 142 to the outlet 122 of the pump 120.

9 This embodiment also includes a first valve 150  
10 connected to the aft port conduit 136 for controlling fluid  
11 medium flow between the aft port 134 and the outlet 122 of  
12 the unidirectional pump 120, and a second valve 152  
13 connected between the aft port conduit 136 and the forward  
14 port conduit 132. In this embodiment, the second valve 152  
15 allows fluid medium received in the aft port 134 to be  
16 directed to the inlet 124 of the pump 120.

17 This embodiment having the unidirectional pump 120  
18 includes a third valve 154 connected between the forward  
19 port conduit 132 and the inlet 124 of the pump 120, for  
20 controlling fluid medium flow from the forward port 130 to  
21 the inlet 124 of the pump 120. A fourth valve 156 is  
22 connected between the radial outlet port conduit 144 and the  
23 forward port conduit 132, for allowing fluid medium  
24 discharged from the pump outlet 122 into the radial outlet  
25 port conduit 144 to be directed into the forward port  
26 conduit 132.

1           To provide motion to the undersea vehicle 112, FIGS.  
2 5A-5C, the first, second, third and fourth valves 150-156  
3 are selectively opened and closed. To provide forward  
4 motion in the direction of arrow 160, FIG. 5A, the third  
5 valve 154 is opened so that fluid medium received into the  
6 forward port 130 is passed through the forward port conduit  
7 132 into the inlet 124 of the pump 120. The first valve 150  
8 is opened and the second valve 152 is closed so that the  
9 fluid medium discharged from the outlet 122 is directed to  
10 the aft port 134, causing a thrust that moves the undersea  
11 vehicle 112 in the direction of arrow 160. The fourth valve  
12 156 is closed so that fluid medium discharged from the  
13 outlet 122 of the pump 120 is directed into the radial  
14 outlet port conduit 144. The fluid medium is then  
15 discharged selectively through radial outlet ports 140, 142  
16 to move the undersea vehicle 112 in radial directions or to  
17 control pitch, roll and yaw.

18           To provide reverse motion in the direction of arrow  
19 162, FIG. 5B, the first valve 150 is closed and the second  
20 valve 152 is opened so that fluid medium received in the aft  
21 port 134 is directed through the intermediate conduit 138 to  
22 the inlet 124 of the pump 120. The pump 120 then discharges  
23 the fluid medium through the outlet 122 and into the radial  
24 outlet port conduit 144. The fourth valve 156 is opened so  
25 that a portion of the fluid medium discharged into the  
26 radial outlet port conduit 144 is directed to the forward  
27 port conduit 132 and discharged out of the forward port 130,

1 causing the undersea vehicle 112 to move in the reverse  
2 direction indicated by arrow 162. The third valve 154 is  
3 closed to prevent the fluid medium being discharged through  
4 the forward port conduit 132 from being fed back to the pump  
5 inlet 124.

6 To provide a hovering motion (no forward or reverse  
7 motion, generally in the direction of arrows 164, 166) the  
8 first valve 150, FIG. 5C, is closed, the second valve 152 is  
9 opened and the third valve 154 is opened so that fluid  
10 medium received in both the forward port 130 and aft port  
11 134 is directed to the inlet 124 of the pump 120, thereby  
12 nulling the forward or reverse motion of the undersea  
13 vehicle 112. The fluid medium is then discharged to the  
14 radial outlet port conduit 144 to the radial outlet ports  
15 140, 142. As described above, the discharge of the fluid  
16 medium through each radial outlet port can be selectively  
17 controlled to vary the depth of the undersea vehicle 112 or  
18 change the pitch or yaw of the undersea vehicle 112 while  
19 hovering. The fourth valve 156 is closed to prevent the  
20 fluid medium being discharged through the radial outlet port  
21 conduit 144 from being directed to the forward port conduit  
22 132.

23 Accordingly, the undersea vehicle propulsion and  
24 attitude control system of the present invention controls  
25 the movement of an undersea vehicle in multiple planes, e.g.  
26 forward motion, reverse motion, hovering, pitch, roll and  
27 yaw, using only a single reversible or unidirectional pump.

1 The propulsion and attitude control system of the present  
2 invention thereby reduces the noise generated when moving  
3 and changing directions of the undersea vehicle, reduces the  
4 amount of space required, reduces the weight of the undersea  
5 vehicle as a whole, reduces the cost of the system and  
6 allows quicker changes in direction and force.

7 In light of the above, it is therefore understood that  
8 the invention may  
9 be practiced otherwise than as specifically described.

1 Navy Case No. 77994

2

3 UNDERWATER VEHICLE PROPULSION AND ATTITUDE CONTROL SYSTEM

4

5 ABSTRACT OF THE DISCLOSURE

6 An undersea vehicle propulsion and attitude control system  
7 is used to control the forward/reverse movement, vertical up/down  
8 movement, lateral movement, pitch, roll and yaw of an undersea  
9 vehicle. The propulsion and attitude control system includes a  
10 forward port at a forward end of the undersea vehicle, an aft  
11 port at an aft end of the vehicle, and radial ports extending  
12 radially along the undersea vehicle. The propulsion and attitude  
13 control system further includes a single pump, either reversible  
14 or unidirectional, and a plurality of valves that, through a  
15 controller, selectively control fluid flow between the pump and  
16 the forward port, aft port, and radial outlet ports in the  
17 undersea vehicle, to provide accurate vehicle propulsion and  
18 attitude control.

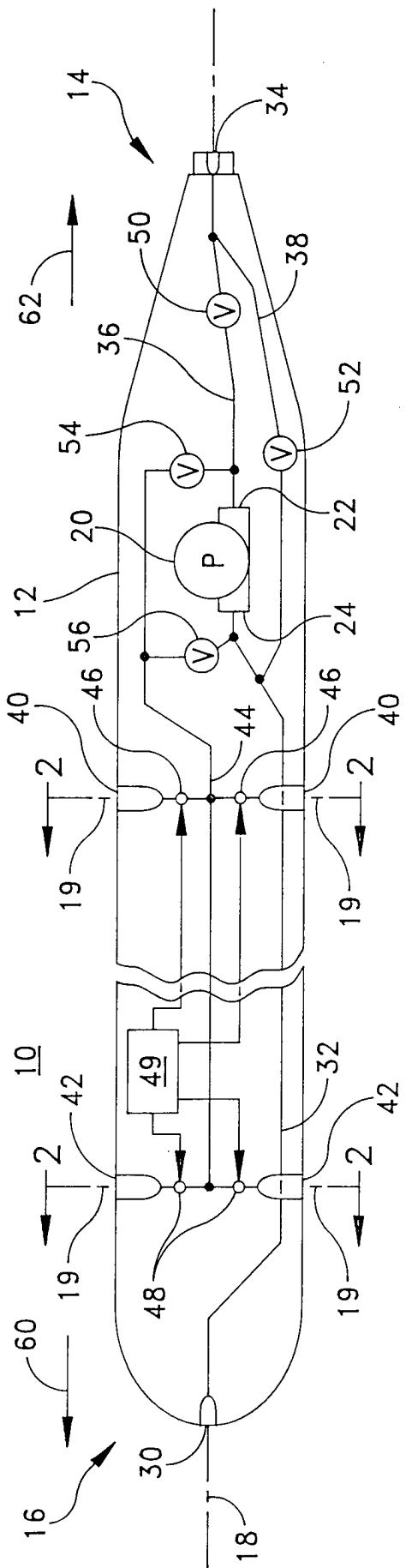


FIG. 1

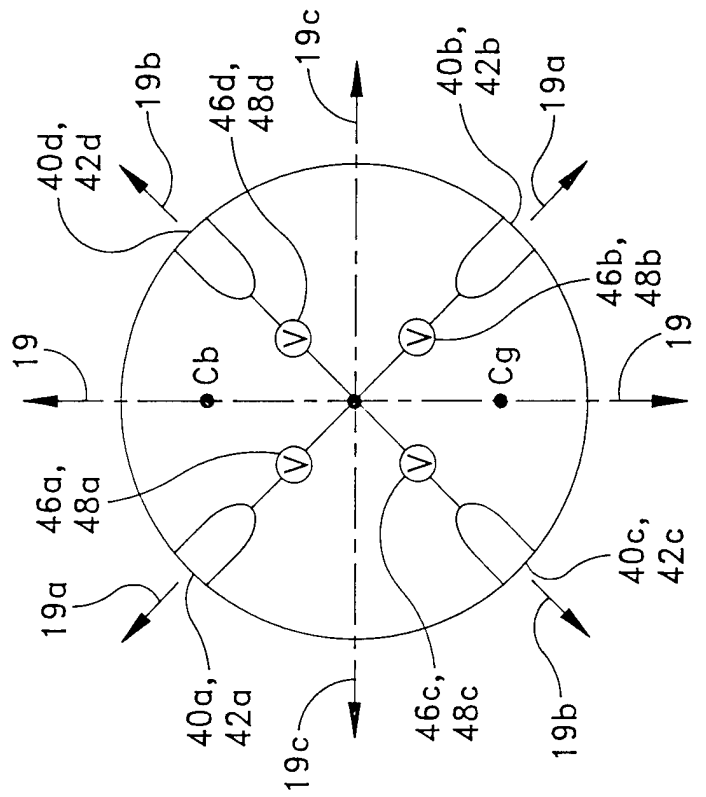


FIG. 2

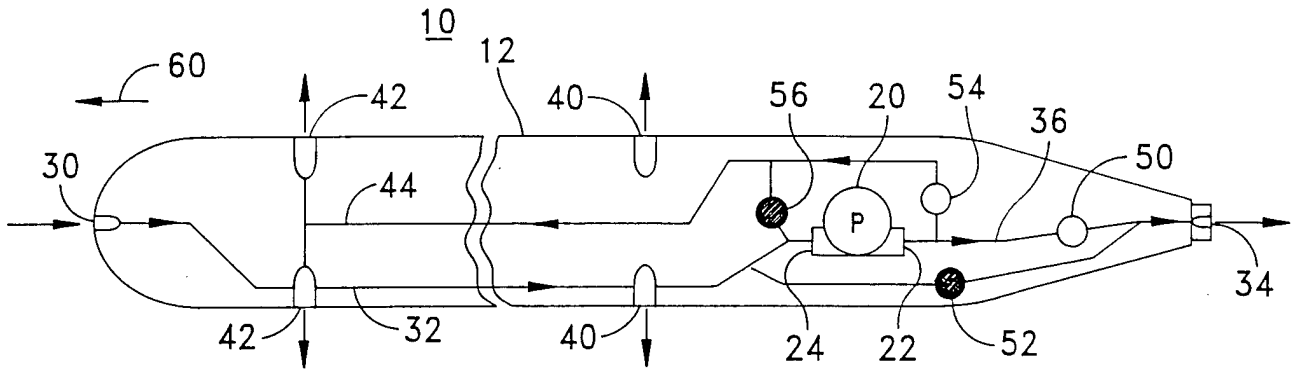


FIG. 3A

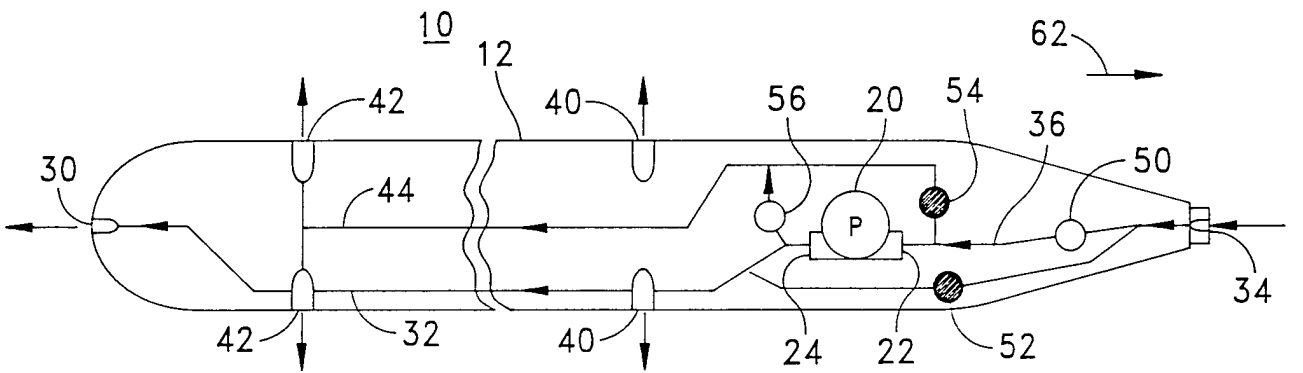


FIG. 3B

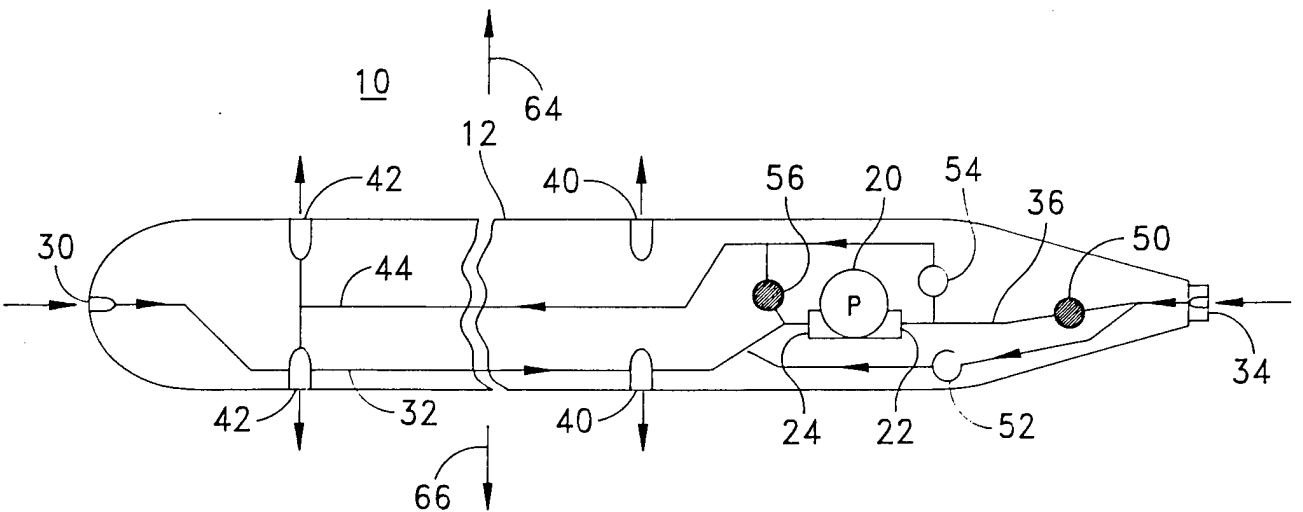


FIG. 3C

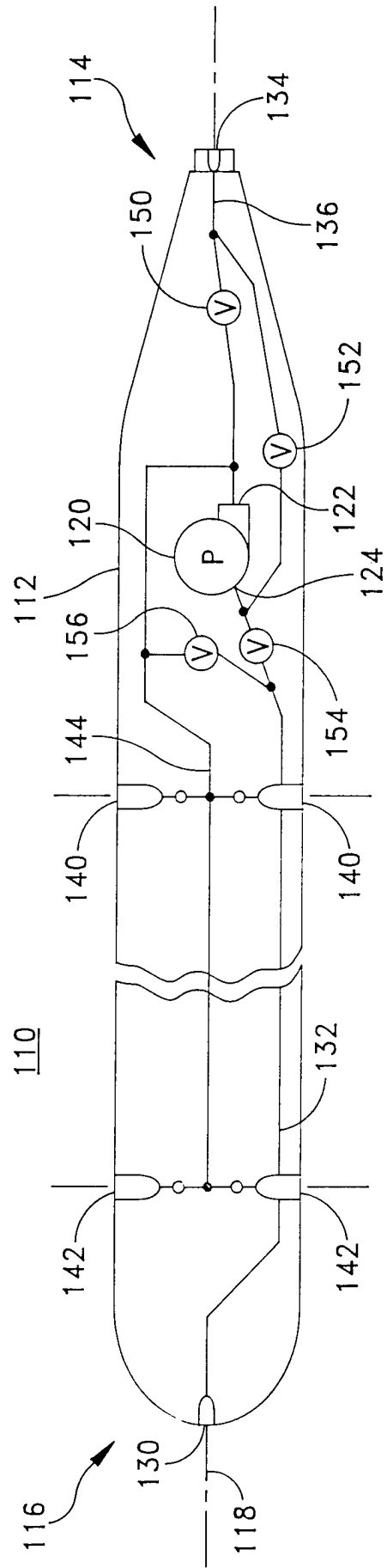


FIG. 4



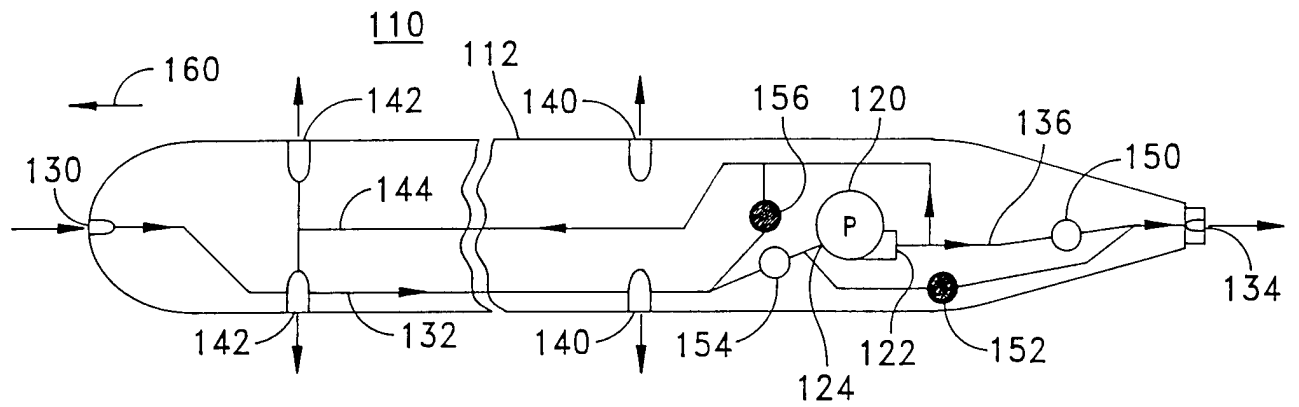


FIG. 5A

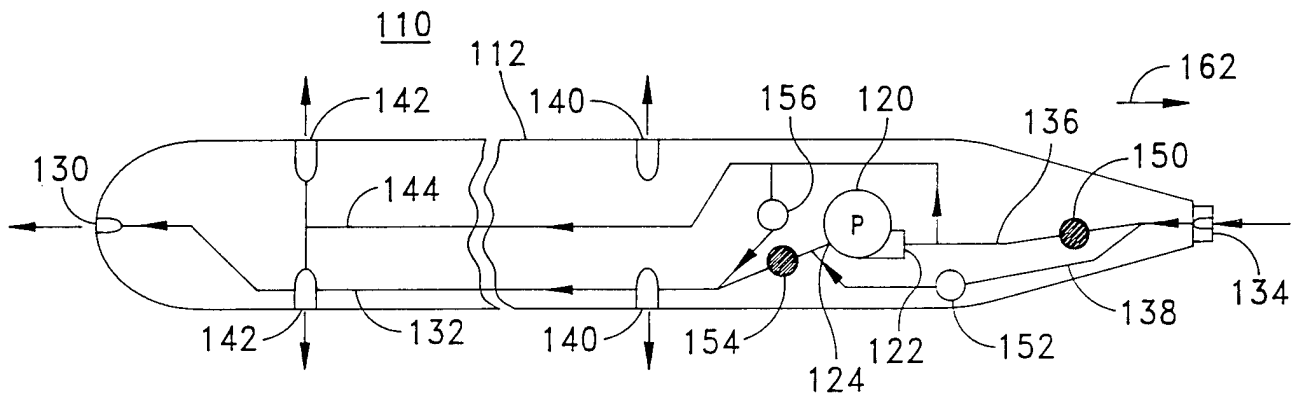


FIG. 5B

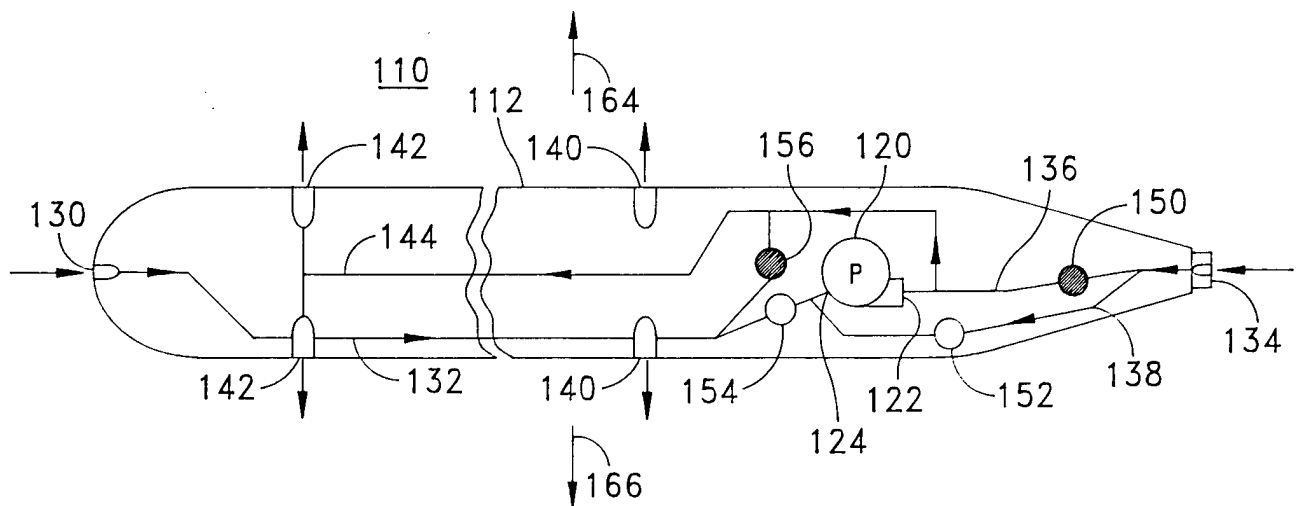


FIG. 5C