

**FUEL SYSTEM DURABILITY—
U.S. COAST GUARD**

**INTERIM REPORT
TFLRF No. 393**

by
Douglas M. Yost

**U.S. Army TARDEC Fuels and Lubricants Research Facility
Southwest Research Institute® (SwRI®)
San Antonio, TX**

for
**U.S. Army TARDEC
Force Projection Technologies
Warren, Michigan**

**Contract No. DAAE-07-99-C-L053 (WD20)
SwRI Project No. 08.03227.20.001**

Approved for public release: distribution unlimited

May 2008

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**Steven D. Marty, Director
U.S. Army TARDEC Fuels and Lubricants
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14. ABSTRACT This project attempted to determine the fuel lubricity requirements of the Sulzer 12ZAV40S fuel injection system. Wear rates with F-76 and JP-5 fuels were attempted using radioactive tracer technologies using a fuel injection system test stand. Within the sensitivity of the resulting activation, and due to detector instabilities, the wear of fuel injection components could not be quantified using on-line in-situ radioactivity monitoring techniques. Off-line monitoring did indicate wear, but only after a catastrophic wear event. The methods did indicate however that the wear with JP-5 fuel was not significantly different than Diesel Fuel Marine (DFM) fuel within the scope of the method with the high durability components. Recommendations were made for future testing.					
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radioactive tracer		surface layer activation		HFRR	
contamination		scuffing load		scanning electron imaging	
JP-5		wear		cavitation	
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EXECUTIVE SUMMARY

Problems: The ability to operate Wartsila/Sulzer 12ZAV40S engines in the US Coast Guard on JP-5, and other alternate fuels, without fuel injection system durability impact is in question. Data from the manufacturer on usage with alternate fuels has not been made readily available.

Objective: To determine the fuel lubricity requirements of the Sulzer 12ZAV40S fuel injection system.

Importance of Project: The U.S. Coast Guard (USCG) prefers to obtain fuel from Navy bunkers, primarily because F-76 has a stability requirement versus MGO, which does not have a stability specification. The Navy is considering a one-fuel strategy, with the one-fuel being JP-5 grade aviation turbine fuel. Thus, the USCG wants to understand the impact of JP-5 on the fuel injection equipment in large medium-speed diesel engines.

Technical Approach: Meaningful wear information can be acquired in short periods of operation using radioactive tracer component activation methodologies. The wear rate with F-76 was to be determined along with JP-5 for comparison of relative component life. A recirculating fuel system was utilized to improve measurement accuracy and to reduce fuel requirements

Accomplishments: A unit fuel injection pump test rig was designed and built for operating the fuel injection system from one cylinder of a Sulzer 12ZAV40S diesel engine. A Surface Layer Activation radioactive tracer experiment was designed for measuring wear rates between fuels. Within the sensitivity of the resulting activation, and due to detector instabilities, the wear of fuel injection components could not be quantified using on-line in-situ radioactivity monitoring techniques. Off-line monitoring did indicate wear, but only after a catastrophic wear event. The methods did indicate however that the wear with JP-5 fuel was not significantly different than Diesel Fuel Marine (DFM) fuel within the scope of the method with the high durability components (24,000-hour).

Recommendations: Due to complications the radioactive tracer approach was not suitable to determine wear in the 12ZAV40S fuel injection system. Some type of controlled durability test would be required to determine suitability of JP-5. Indicators suggest short term JP-5 usage would likely be suitable, however evidence of cavitation erosion suggest long term use may be questionable. However the cavitation erosion seen was not documented for both fuels examined, nor is it certain expanded speed operating conditions may have contributed to accelerating the cavitation erosion. Two approaches are suggested for determining wear with JP-5:

- A controlled test of an operating engine where one cylinder is isolated to operate on JP-5 for a significant number of hours. Component condition would be compared to one of the other cylinder conditions.
- A single pass pump rig test of a significant quantity of JP-5 fuel, such that the effects of additive changes and fuel/oil migration are not compounded. The wear on the barrel and plunger would be compared to an installed plunger removed from the Healy that has been burning DFM.

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The U.S. Army TARDEC Fuel and Lubricants Research Facility (TFLRF) located at Southwest Research Institute (SwRI), San Antonio, Texas, performed this work during the period September 2002 through January 2008 under Contract No. DAAE-07-99-C-L053. The U.S. Army Tank-Automotive RD&E Center (TARDEC), Force Projection Technologies, Warren, Michigan, administered the project. Mr. Luis Villahermosa (AMSRD-TAR-D, MS110) served as the TARDEC contracting officer's technical representative, and Mr. Thomas Gahs, United States Coast Guard, served as the project technical monitor.

The author would like to acknowledge the assistance of Messrs. Douglas Eberle, Rodney Grinstead, Max Reinhard, Jr., Kenneth Ellebracht, Jay Johnson, and Daniel Anctil for constructing and conducting the testing and Ms. Rebecca Emmot for editing and processing the report.

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ACRONYMS AND ABBREVIATIONS

°C	degrees Centigrade
°F	degrees Fahrenheit
µm	micrometer, 1×10^{-6} meter
BSE	Back-Scattered Electrons Ca Calcium
cSt	centiStoke
DFM	Diesel Fuel Marine
HFO	Heavy Fuel Oil
HFRR	high-frequency reciprocating rig
HP <i>or</i> hp	horsepower
L	liter(s)
MeV	Mega electron volt
MGO	marine gas oil
MILSPEC	military specification
min	minute(s)
NaI	Sodium Iodide
ppm	parts-per-million
rpm	rotation(s) per minute
SED	Secondary Electron Detector
SEM	Scanning Electron Microscope
SLA	Surface Layer Activation
SwRI	Southwest Research Institute
USCG	U.S. Coast Guard
ULSD	ultra low sulfur diesel
WSD	wear scar diameter

1.0 INTRODUCTION AND BACKGROUND

The ability to operate Wartsila/Sulzer 12ZAV40S engines in the U.S. Coast Guard (USCG) on JP-5, and other alternate fuels, without fuel injection system durability impact is in question. Data from the manufacturer on usage with alternate fuels has not been made readily available.

2.0 OBJECTIVE

The objective of this project is to determine the fuel lubricity requirements of the Sulzer 12ZAV40S fuel injection system. The USCG prefers to obtain fuel from Navy bunkers, primarily because F-76 has a stability requirement versus marine gas oil (MGO), which does not have a stability specification. The Navy is considering a one-fuel strategy, with the one-fuel being JP-5 grade aviation turbine fuel. Thus, the USCG wants to understand the impact of JP-5 on the fuel injection equipment in large medium-speed diesel engines.

3.0 APPROACH

The initial approach was to obtain information on the Sulzer 12ZAV40S engine fuel injection components and determine the likelihood of durability issues with alternate fuel usage. Alternate fuels considered other than F-76 will be JP-5 and ultra low sulfur diesel (ULSD) fuels. This effort was accomplished by coordinating with Wartsila/Sulzer on alternative fuel usage, to determine the level of information available, fuel acceptance, and willingness to participate in a test program.

Based on the available alternative fuels usage information, a decision was made to design and build a stand to drive Sulzer 12ZAV40S fuel injection system components. The stand was developed because the results of the fuel usage experience survey suggested testing needed to be performed (i.e. data not available from Sulzer) in order to assess fuel lubricity affects.

Meaningful wear information can be acquired in short periods of operation using radioactive tracer component activation methodologies. The wear rate with F-76 was to be determined along with JP-5 for comparison of relative component life. A recirculating fuel system was utilized to improve measurement accuracy and to reduce fuel requirements.

4.0 FUELS USAGE

Efforts were made to establish contacts with Wartsila on the Sulzer engines. Efforts were made to identify the manufacturer of the Sulzer ZA40S fuel injection system. Mr. Tapio Markkula of Wartsila was contacted about Sulzer engines and the use of JP-5. He was aware of a Wartsila engine power plant running JP-4 in Alaska, due to unheated fuel storage, without any injection system problems. Mr. Markkula indicated the Wartsila engine burning JP-4 used L'Orange injection systems, and suggested Sulzer also used L'Orange fuel systems. Mr. Markkula supplied Southwest Research Institute (SwRI) with the Wartsila representative for the USCG Healy, Mr. Arne Vestad. The U.S. service center for L'Orange fuel systems were contacted, Fluid Mechanics Inc.. Mr. Kerry Cervenec at Fluid Mechanics Inc., Ft. Lauderdale, FL location indicated that although Sulzer engines do use L'Orange systems, the ZA40S series fuel injection systems are manufactured by DUAP, a Swiss company. Fluid Mechanics Inc. is also the U.S. representative for DUAP. Mr. Cervenec offered to solicit both L'Orange and DUAP for information on kerosene fuels in marine fuel injection systems for medium-speed diesels, but there was not any information forwarded.

Mr. Georges Bigand, Service Engineer for Wartsila North America in Seattle contacted SwRI and indicated that Wartsila does not have any data available on wear with low viscosity and low lubricity fuels, especially with fuels such as JP-5. It was indicated that no engine was known to be running with JP-5, thus a comparison with Heavy Fuel Oil (HFO) would be difficult. Mr. Bigand indicated that no trials in the engine laboratory were foreseen for the near future on fuel lubricity with Sulzer Z40S. It was indicated that there were never any negative reports about fuel injection systems operating on Marine Diesel Oil.

The manufacturers of the Z40S fuel injection systems were confirmed to be both OMT and DUAP. It was indicated that wear due to cavitation is possible if the fuel oil system is not properly adjusted (1)*, but there were no worn components available for inspection at Seattle or WCH (Wartsila Switzerland in Winterthur).

A Wartsila technical service manager in Italy indicated that attention should focus on the fuel pump barrel and plunger clearance, which should preferably be smaller with kerosene fuels than HFO engines. With the very low viscosity of JP-5, and the clearances for the plunger and barrel calculated for HFO, one has to look at possible leakage down the plunger, which may induce lubricating oil contamination with fuel.

SwRI has been following up with the Wartsila contacts for the Sulzer ZA40S engines and also attempting to establish contacts with the fuel injection component manufacturers DUAP and OMT. DUAP indicated good experience on other engines using critical fuels with coatings at the injection pump plunger and the injection nozzle together with special heat treatment. Depending on the results of testing, this could be an option for the Healy. DUAP indicated they could make special test components if required.

5.0 FUEL INJECTION PUMP TEST FACILITY

The fuel injection pump test facility would require a cam support, camshaft components, and fuel injection system components. Due to the low rpm, large injection quantities, and high injection pressure a sizeable flywheel was required to operate the cambox. A flywheel is required to minimize the speed variations of the injection cam during the injection event. Preliminary calculations suggest 15-hp will be required to inject the fuel for one injector. More accurate injection pressure and delivery data was obtained to refine the cam box design.

The drive system for the injection pump was designed. A motoring dynamometer with an electromagnetic clutch was to be used to drive the injection system. The clutch and drive system was verified to operate with stability and controllably at 257 rpm, the expected Sulzer engine

fuel injection pump speed. Injection system data was reviewed and it was determined a single flywheel of appropriate mass, shafts, and flywheel support bearings taken from a single cylinder engine crankcase would provide the requisite inertia. It was calculated the single cylinder crankcase crankshaft support bearings would support the Sulzer fuel cam with a sufficient safety factor. Modifications to the crankcase to mount the injection pump were determined from cam dimensions provided by Wartsila. The test cell stand was configured and modified for the flywheel, bearings, and shafts and the coupling between the motoring dynamometer and injection stand was machined.

The engine technical manuals were received, along with engine serial numbers, to determine component part designations (2,3,4,5,6,7). Information obtained from the engine tech manuals was used to layout the design of the test stand and the test loop fuel system. A source of engine components was located, and quotes for parts obtained. A Wartsila representative supplied the Healy component designations in order to obtain the most current parts. The fuel injection parts as installed on the Healy are shown in Table 1. The fuel injection parts as installed on the Healy were used to issue a purchase requisition.

Table 1. Healy Main Engine Injection Specification

Healy Main Engine Injection Specification					
Engine		740386	740387	740388	740389
Originaly	Injectors	144° X 10 X	144° X 10 X	144° X 10 X	144° X 10 X
	Fuel Delivery Pipe Ø)	6 mm	6 mm	6 mm	6 mm
	Barrel/Plonger	C.I.T*	C.I.T*	C.I.T*	C.I.T*
	Effective Delivery	14.73 mm	14.73 mm	14.73 mm	14.73 mm
Present	Injectors	160° X 13 X 0.60	160° X 13 X 0.60	160° X 13 X 0.60	160° X 13 X 0.60
	Fuel Delivery	7 mm	7 mm	7 mm	7 mm
	Barrel/Plunger	VD 2**	C.I.T*	VD 2**	VD 2**
	Effective Delivery	14.73 mm	14.73 mm	14.73 mm	14.73 mm
* C.I.T : Constant Injection					
** VD 2 : Variable Damping 2nd					

The parts purchased included the two-piece bolt-on replacement fuel injection cam shown in Figure 1 below. The unit fuel injection pump shown in Figure 2 is for one cylinder of the Sulzer 12ZAV40S engine. Figure 3 is the fuel injection line that transmits high-pressure fuel from the injection pump to the delivery valve holder. The delivery valve holder and injection line that mounts into the cylinder head, is shown in Figure 4. The fuel injector, nozzle holder and the nozzle, are shown in Figure 5. The final components purchased were a spare fuel injection pump barrel and plunger as shown in Figure 6. The barrel and plunger set will be sent for exposure to a radioactive isotope. The plunger will be removed from the barrel and exposed in the area of the fuel control helix.



Figure 1. Fuel Injection Cam for 12ZAV40S Engine

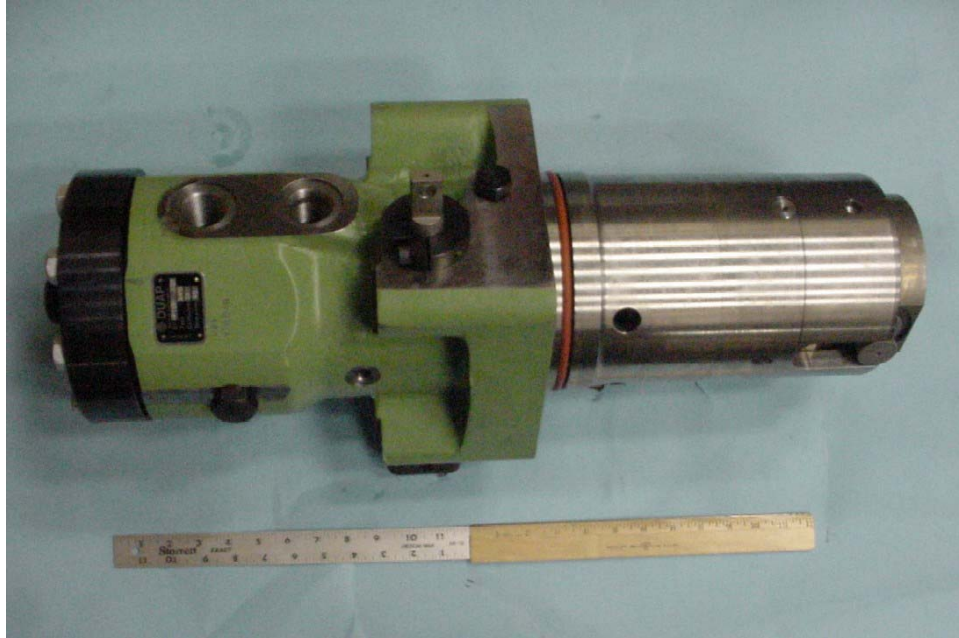


Figure 2. Unit Fuel Injection Pump for 12ZAV40S Engine



Figure 3. Injection Line for 12ZAV40S Engine



Figure 4. Delivery Valve Holder and Injection Line for 12ZAV40S Engine



Figure 5. Fuel Injector Nozzle Holder and the Nozzle for the 12ZAV40S Engine



Figure 6. Barrel and Plunger from an Injection Pump for the 12ZAV40S Engine

After receipt of the test components it was determined the single-cylinder crankcase was too narrow to accommodate the two-piece replacement cam lobe. The two-piece cam was substantially larger than anticipated from the standard cam dimensions that had been provided by Wartsila. The preliminary injection box design needed to be resized to fit the larger two-piece cam. Therefore a cam box was designed to support the cam lobe and camshaft. The cam box was built with steel plates, bearings, and shafting shown in Figure 7. Machining and fabrication of the cam box included the align boring of the bearing bores and the machining of the injection pump mounting surface for perpendicular alignment. The cam box components shown in Figure 8 were assembled and measured for final dimensional adjustments for installation of the fuel injection pump. The final adjustments were made from installed height data provided by Wartsila. A fuel injector-mounting stand was fabricated to hold the injector, high-pressure fuel line and return line in the proper orientation.

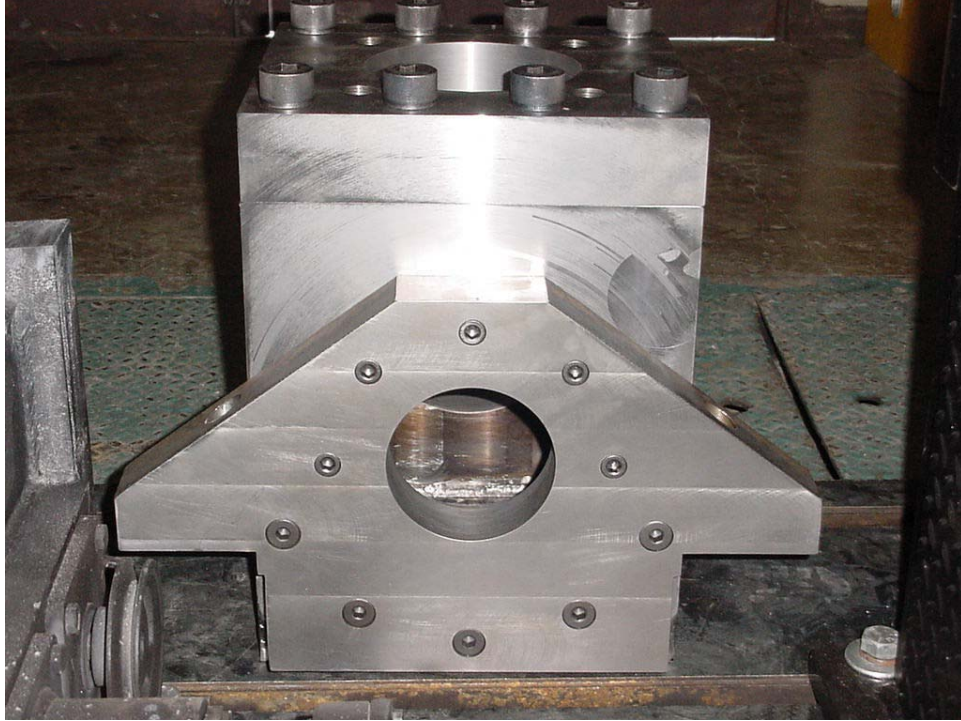


Figure 7. Cam Box for 12ZAV40S Fuel Injection System



Figure 8. Cam Element, Shafting, and Bearings for 12ZAV40S Components

The eddy-current clutch of the motoring dynamometer drive system was proven to be a mismatch for the required speed of the fuel injection pump. The current demand on the drive motor is excessive above ½-rack position. The drive option that will have the best match of speed and torque demand is a hydraulic drive system for the fuel injection pump. SwRI had a hydraulic powerpack available for use by the project. A low speed, high torque hydraulic motor was attained, along with a hydraulic pressure control valve to attain proper speed control. The available hydraulic power pack and controllers were removed from storage, and seals and components were reconditioned. The electric motor, hydraulic pump, hydraulic motor, motor mount, fluid reservoir, heat exchanger, couplings and hydraulic lines were installed for the hydraulic drive. The completed fuel injection system stand is shown in Figure 9.



Figure 9. Assembled 12ZAV40S Engine Unit Injection Pump Stand

6.0 RADIOACTIVE TRACER EXPERIMENTAL DESIGN

A test program was outlined for radioactive tracer wear measurements. The test program was to establish a break-in level with Diesel Fuel Marine (DFM), than perform wear rate measurements with DFM, followed by JP-5. The wear rates for JP-5 and DFM were to be repeated several times to establish consistency and statistical information. The fluid test loop was designed using injector flows and bypass rates at the rated operating condition to finalize the test loop design.

Preliminary calculations suggest that because of the anticipated low wear rates and the desire to obtain measurable wear debris signal in the test fluid, a bulk activation technique would not be appropriate. The anticipated in-service life of the fuel injection' components are 24,000 hours per the Technical Manual. A contact was established with DUAP, an injection component supplier for the ZA40S series engines. The injection pressure of this engine is about 1200 bar and the injection pump plunger has a diameter of 35 mm and a delivery stroke of 15.9 mm. At this size, the Surface Layer Activation (SLA) radioactive tracer method is the only safe method to use. The most critical components reported during operation with low viscosity and/or high corrosive fuels are the injection pump plunger with cylinder and the injection nozzle shaft. Due to the size of the ZA40S components, bulk activation would result in components that are radioactively too contaminated to be handled safely within human exposure limits.

SLA techniques were required. The SLA technique radiates a small area, a few microns deep, then monitors the reduction in activity as the part wears. SLA requires knowledge of the location of greatest wear, so the component can be activated in that area. The area of wear of the injection components was sought from the component manufacturers to improve credibility of the wear data generated. SLA also requires a detailed calibration procedure that establishes the material removal versus activity curve. The SLA test procedures were outlined, and resources for activation and calibration established.

The activation design is proved to be a challenge due to the mass of metal that shields the detector from the activated injection plunger. Current thought is a shallow grazing angle will be required for the activation to get the measurement sensitivity needed. However a shallow grazing angle is also affected by the surface finish. Surface finish was determined and estimates were

made to determine the activation profile needed to provide reasonable wear resolution within several hours of operation. The activation of the components took place after the cam box was completed and the injection pump and fuel system were operational so that the test components could be run-in.

SLA works by developing a master calibration curve of radiation versus material removal. Because of the cost of the components, a test coupon was used to develop the calibration curve. DUAP had supplied the designation of the fuel injection pump plunger alloy. SwRI received a report from Chuck Blatchley at Pittsburg State University describing the results for a trial activation. They bombarded four samples with 6 MeV protons at an incidence angle of 60°. They then performed depth calibration measurements using a polishing technique to produce curves of activity as a function of depth. Based upon these curves, they developed a fourth order polynomial equation describing wear as a function of counting rate. They also measured the isotope production rate of the activation. SwRI believes that 10 microcuries of Co-56 are needed within 7 μm of the surface to accurately make this measurement. Although the activation depth was right on target, the production rate achieved by Pittsburg state was not sufficient to reasonably produce this much activity (8).

As an alternative, SwRI approached Ken Oxorn of ANS Technologies in Montreal, Canada to perform the activation. SwRI had worked with ANS Technologies in the past and believed he can perform a reliable activation. Rather than perform a physical depth calibration, Ken uses a theoretical model to determine relative activity as a function of depth. Based upon his experience he has good confidence the model will be within 2% of the actual activation.

In addition, ANS Technologies had agreed to perform a trial activation at their cost to validate the equipment setup, masking scheme and production rate. SwRI had sent the spare Sulzer plunger to ANS for this activation. The plunger manufacturer DUAP supplied a sketch (Figure 10) of two characteristic wear areas on the fuel injection pump plunger. One wear area was at mid-stroke on the plunger most likely where maximum fuel injection pressure is developed. ANS made a tantalum mask for the plunger in the area designated by SwRI for the trial activation. The activation was performed and the preliminary results indicated the masking and production rates were satisfactory.

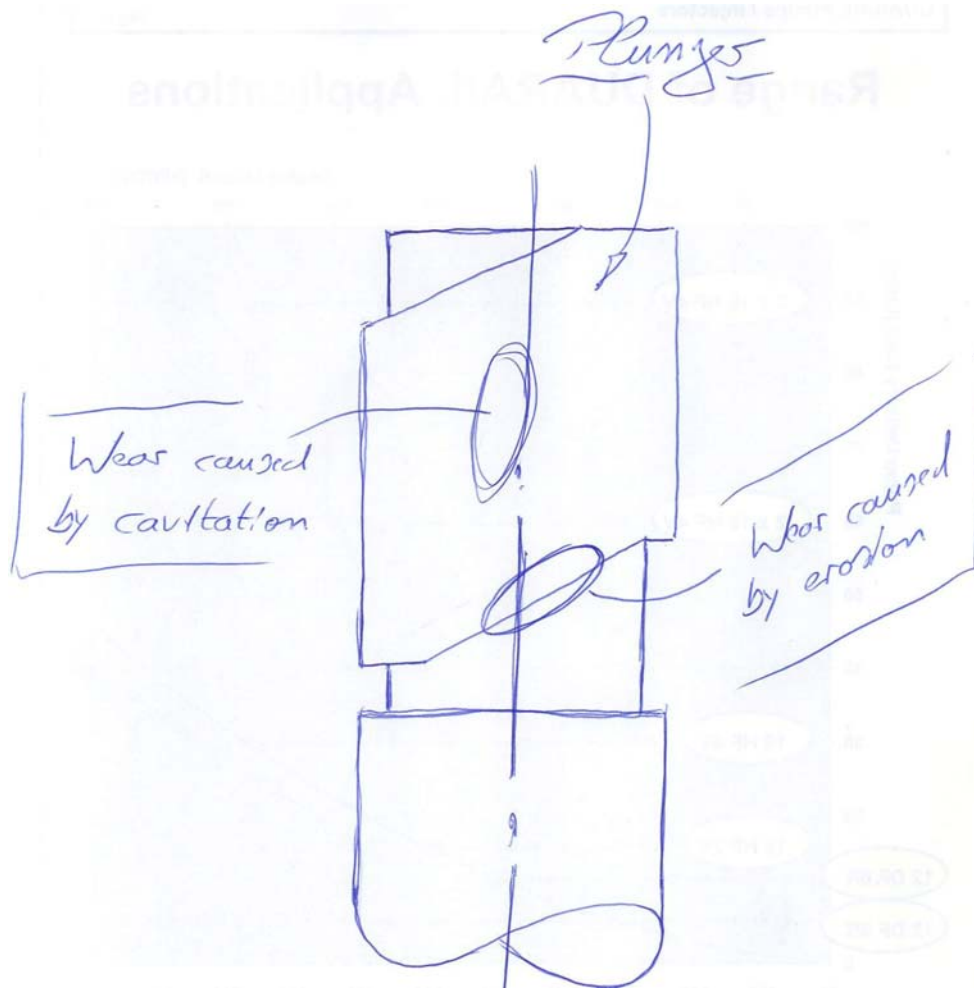


Figure 10. Sketch from DUAP on Plunger Wear Areas

The activation of the test components took place after the fuel injection pump was operated for 20 hours to run-in the wear surfaces using a 3000-g lubricity level fuel. The included figures show radiographs of the plunger intended for wear testing and the plunger used for the trial activation. The intention was to activate two areas on the plunger, 180 degrees apart, with about a total of 10 microcuries of Co-56. Figure 11 shows the test plunger, which showed 36 microcuries of Co-56 activity, initially attributed to a good beam. Figure 12 is a plot of the targeted activity versus depth profile for the activation.

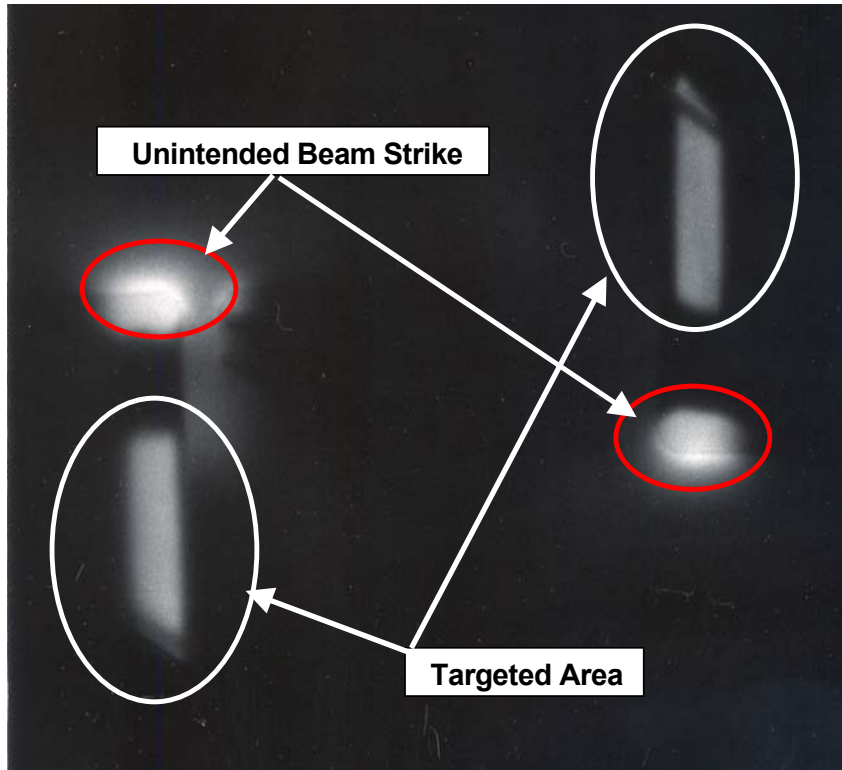


Figure 11. Radiograph Showing Target Area and Unintentional Activation

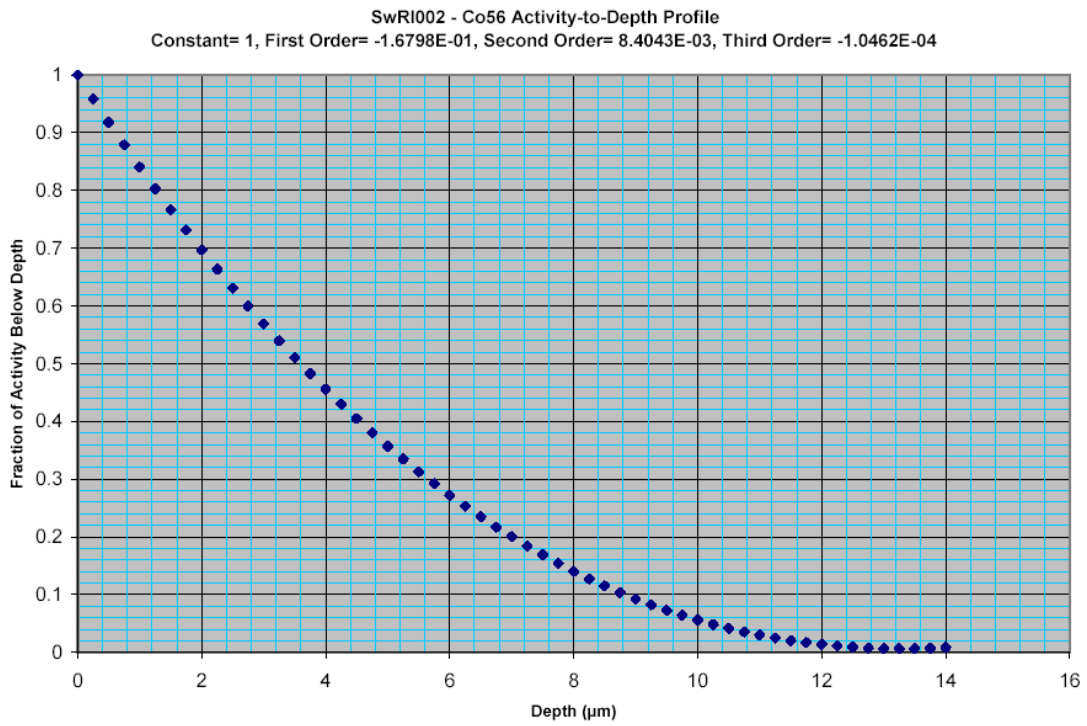


Figure 12. Activity Versus Depth Profile for Co-56

Examination of the radiograph indicates the activity is high because the beam struck an unmasked vertical edge of the plunger. It is likely most of the activity measured is on the vertical surface, normal to any wear surface. Figure 11 does show activity in the target area, but SwRI cannot separate the activity from that area with the detectors. Furthermore, the activity depth profile is unknown due to the beam strikes on the different surfaces. It appears the mask was not extended far enough along the length of the plunger to block the beam from an unintentional strike (9). The subcontractor acknowledged their error, and will activate another plunger at no cost.

Figure 13 shows the trial plunger, which had only 3 microCuries of Co-56 (which is too low an activity level to get good measurements) in a single target area. In this case the mask was applied properly and there are not any stray areas of activity. SwRI used the original trial plunger after running a 20-hour break-in for activation at ANS Technologies expense. Additional activation was successfully added to the original activation at the areas desired. Figure 14 shows the successfully activated trial plunger, which resulted in 35 microCuries of Co-56 that was used for testing.



Figure 13. Radiograph of Targeted Area on Trial Activation Plunger



Figure 14. Successfully Activated Injection Plunger

7.0 TESTING

Operation of the test apparatus was initiated with the 3000-g scuffing load fuel, a surrogate fuel for DFM, to establish a baseline wear rate. A test fixture was mounted on the fuel injection pump that precisely held a gamma detector at the same location. The detector was kept at a fixed vertical and radial location when activity counts were taken. Likewise the flywheel of the fuel injection pump was always returned and locked into a peak lift position when counts were taken so that the activated area of the plunger was always in the same spot. A hardened deflection screw was removed from the side of the fuel injection pump, and the fuel rack was turned so that the activated area had a visually clear path through the fill/spill port and deflection screw bore to the detector.

SwRI operated the fuel injection pump at 257 rpm and 3/4-rack over a period of 35 hours, taking activity readings at approximately 4-hour intervals on the 3000-g fuel. SwRI initially used two Ludlum Model 1000 Scalar counters with Model 44-2 gamma probes to measure gamma

emissions from the test plunger. The initial test protocol was to take a series of ten 1-minute measurements on a background, a reference, the test article and a reference repeat. It was known that these detectors have a temperature dependant drift. The initial plan was to use the reference measurements to correct for this drift. As a reference, SwRI used the plunger that had been irradiated in the wrong spot.

Subsequent testing revealed that the reference measurements obtained were not constant, thus making the test plunger measurements unreliable. Suspecting a problem with the Ludlum gamma probes, a switch was made to a three inch NaI detector with a computerized spectroscopy system. This allowed viewing counts as a function of energy. Initially the detector was set up on the reference plunger. When the detector was put in the vicinity of the test article, however, the energy peaks shifted significantly. Investigation revealed that the NaI gamma probes are sensitive to magnetic fields. Measurements made around the injection pump assembly with a magnetometer that revealed the presence of a magnetic field. This magnetic field likely changes over time, resulting in sensitivity shifts in the radiation detection equipment.

SwRI fabricated a magnetic shield for the detector and initial tests revealed that this stopped the energy peak shifts when the detector was put near the test article. The detector was set up to count one-minute samples over a weekend and recorded a slow period drift that followed outdoor temperature. Though the reference measurements could be used to correct for this, SwRI felt that a temperature control system could be implemented to minimize this drift and get more consistent measurements. A temperature control system was added and one-minute counts were taken overnight. The overnight measurements proved stable.

Unfortunately data from the 35-hour operation was unusable because it could not be corrected for the magnetic interference. Efforts to mitigate the effects of the magnetic fields and temperature initially appeared to have been successful. SwRI continued measuring the change in activity of the plunger as a function of operating hours. Between 35 and 47 hours the measurement remained nearly constant, following decay correction. This indicated that the plunger was not wearing sufficiently for us to detect a change over this period. The fuel was changed and a significant increase in plunger activity was measured. Since this is not physically

possible, this was an indication that SwRI was still experiencing a detector stability problem. A decrease in activity was recorded between running hours 47 and 66 on the plunger, however due to detector instability, this data is not statistically reliable. This data is shown in Figure 15, with counts adjusted for isotope half-life.

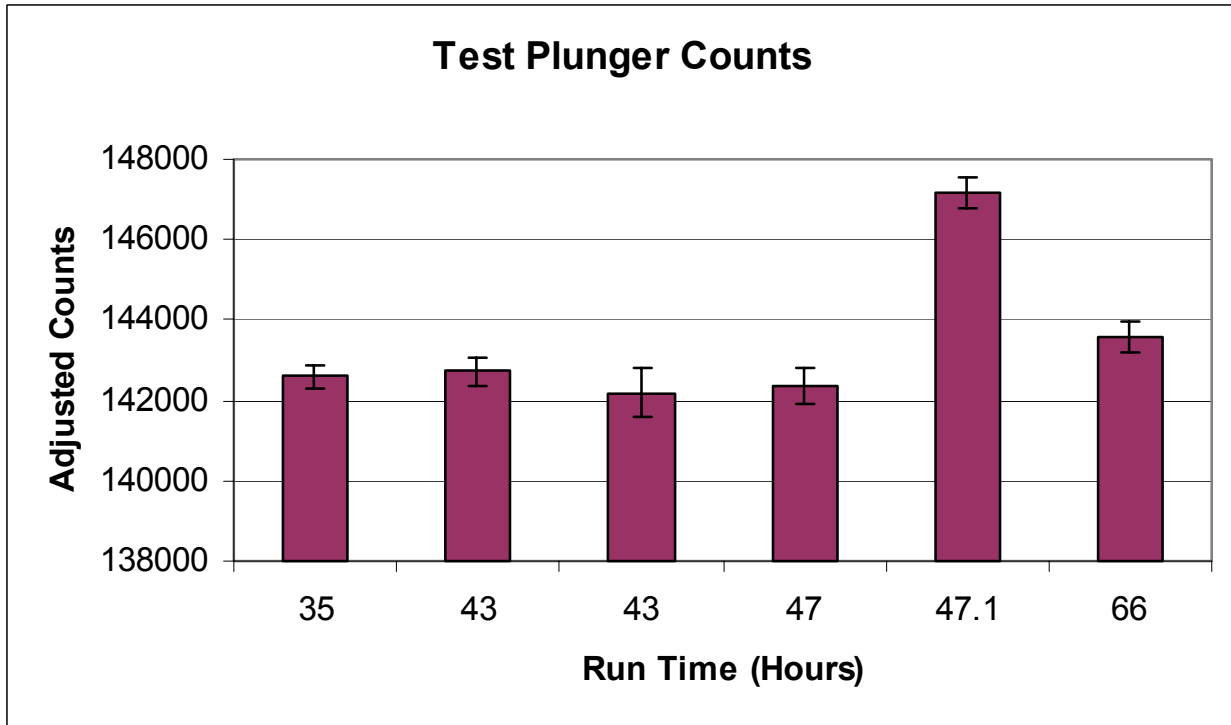


Figure 15. Plunger Activity Adjusted for Half-Life

SwRI suspected there was still a magnetic field effect, which caused fluctuation of the detector readings during in-situ plunger activity measurements. SwRI re-measured the 66-hour data point after a five-day delay and the measurement was statistically the same. The flywheel was then rotated five degrees and the plunger was re-measured. SwRI believes that this rotation was within the dwell location of the plunger, so the plunger should not have moved vertically or rotationally because the rack was fixed. A significant drop in counts was detected. SwRI suspects this may be due to a change in the magnetic field position of the flywheel. SwRI has confirmed the magnetic field effect with the NaI detector. The investigation into the fabrication of a mu-

metal shield to eliminate the magnetic field effects resulted in data that could not be used with confidence.

The best approach for stable in-situ measurements would be to setup and calibrate a germanium detector. It would require setting up a detector and a liquid nitrogen dewar on a table to get it as close to the plunger as physically possible. The difficulty is locating it in a position where it will not be moved at all. This would be very difficult given the level of pounding when the system is running. It will also need vibration damping to protect the crystal. Though SwRI won't be running the germanium detector while the rig is running, SwRI doesn't think it can be moved while running and repositioned accurately like was done with the NaI detector.

There were not sufficient funds remaining to further pursue in-situ activity measurements. A cost-effective option for the project was to remove the plunger from the pump and take it to a lab fixture for activity measurement. SwRI fabricated a jig for accurate positioning. Unfortunately there was not a baseline established previously with the germanium detector. The plan was to remove the plunger, measure for activity, replace the plunger, operate the injection pump on the fuel 3000, and then remove the plunger again for re-measurement. If this proved successful, the process would be repeated with the 2000-g scuffing load fuel.

The base 3000-g scuffing fuel had degraded which resulted in improved fuel lubricity (scuffing load value increased). This fuel batch was also the base fuel for a Navy lubricity program, and insufficient quantity existed for both programs. A replacement batch of base fuel was sought; therefore, testing was initiated with the 2000-g scuffing fuel. The injection rig was de-fueled, cleaned, and flushed prior to filling with the 2000-g fuel. The pump cambox was inspected, the cambox oil changed, and the hydraulic drive system checked. An additional set of activity measurements was taken on the plunger prior to re-assembly of the injection pump. The injection pump has been reassembled and the cleaned and flushed fuel system filled with the 2000-g fuel. The pump and drive were checked for proper operation.

The fuel injection rig was operated for 12 hours and then shut down to check the activity of the plunger. The activity measurements, when corrected for decay suggest that 12 hours of operation

did not result in any detectable activity loss on the plunger. At the end of twelve hours a discoloration was noted in the test fuel. The 2000-g fuel is water white to pale yellow, and a light gray to brown tinge was noted. It was felt lubricating oil could be migrating into the metering section of the pump and being injected. Due to the recirculating fuel system, this meant lube oil could accumulate in the fuel supply and improve the fuel lubricity and skew results. A fuel sample was filtered, and the filter residue was examined by x-ray for elemental content. If the contaminant were lube oil, the additive metals would be revealed in certain proportions. The elemental analysis indicated the contaminant was fuel degradation products, most likely from the previous 3000-g fuel. The fuel system was drained and significant effort was extended to clean the system by swabbing all fuel lines and reservoirs with isooctane.

The injection stand was reassembled after 36 hours of operation, and charged with a new batch of 2000-g fuel to continue testing. A corrosion inhibitor additive was added to the fuel to make the fuel more similar to JP-5. Again, after several hours of operation fuel discoloration was apparent. It was believed the contaminant was lubricating oil migrating up from the bottom end of the injection pump. The bottom end of the injection pump is supplied lubricating oil as is the engine, and at the oil pressure prescribed for the engine. A sample of the test fuel was checked for lubricity, both in the tank volume and the injector effluent.

The high-frequency reciprocating rig (HFRR) lubricity result for the tank was 643-microns and for the injector effluent was 583-microns wear scar diameter. Based on lab repeatability at SwRI these results suggest an improvement of the fuel lubricity due to lubricating oil migrating into the injection pump. However, lubricity results of both fuels indicate they are still low lubricity fuels, with the generally accepted range for a good lubricity fuel being 460 to 520-microns wear scar diameter. Combined with the low viscosity of the 2000-g fuel, it is felt the fuel injection equipment is still being challenged with a fairly severe fuel. The test rig was operated for 72 total hours with the activity of the injection pump plunger checked at 12, 24, 36, and then 72 hours. The fuel was changed at 36 hours as noted. Correcting for half-life, there was not any measurable wear detected with the 2000-g fuel. The injection pump wear results on the 2000-g fuel, which were below 0.14-microns of total wear, suggests that the use of JP-5 should not impact vessel operations.

Initial thought was to try the 1000-g fuel and see if measurable wear would be evident with this type of poor lubricity fuel. However, SwRI recently ran the 1000-g fuel in two different systems that resulted in rapid seizures of the injection pumps. SwRI became hesitant to put the fuel in the Sulzer system for fear of seizing the injection pump; the concern being that the 1000-g fuel has both extremely low lubricity and low viscosity, and being in the kerosene boiling range. The resulting question is how likely would it be that a Fischer-Tropsch fuel would be available to the Coast Guard as a JP-5 versus a DFM? Fischer-Tropsch DFM would be the logical choice because it would meet all ULSD specifications, would have zero aromatics, and would have a higher viscosity.

Although there were some problems with the on-line detection with detector stability while running the 3000-g fuel, it was apparent wear was not seen with that fuel and compounded the problem. This fact was verified by the lack of wear detected with the 2000-g fuel. The 2000-g fuel was additive treated to make it a true JP-5 to answer the Coast Guard JP-5 question directly (i.e., Is JP-5 OK?).

It was decided to continue with the 2000-g + additive, or JP-5 fuel, for an additional 72 hours to determine if wear can be detected. Wear would be detected when it exceeds the detection limits imposed by the activation profile and instrumentation. If wear is detected the units would be microns of material removed, and would be reported as micron/hour wear rate. The DCI-4A additive improved the scuffing load of the neat 2000-g fuel to around 2825-g scuffing load. Of interest was the HFRR result for the fuel, which was 645-micron wear scar diameter (WSD), a result that reveals a poor HFRR sensitivity to additives in small quantities.

Several samples of fuel and the lubricating oil were sent for elemental analysis. These samples included fuel directly from the injector, fuel from the 40-gallon fuel reservoir, fuel/oil mix from the injection pump bleed, and a sample of the lubricating oil from the cam box. Table 2 shows the results of the elemental analysis of the samples in units of parts-per-million (ppm). The cambox lubricating oil shows large concentrations of additive elements that were to be used as tracer elements in the fuel. The injector effluent and fuel tank samples did not reveal any additive metal above the detection limit of the instrument and test method, ASTM D 5185. To put the results in perspective, if 1-ppm Ca did exist in the fuel, based on the lube oil Ca level, the overall concentration of lube oil in the fuel would be around 70-ppm. Most surface-active additive elements are put into fuels at the 10 to 20-ppm levels. Data reported by SwRI in Letter Report No. BFLRF-91-007 (10) shows that lubricating oils in fuels are not effective as a lubricity improver unless the concentrations are above 0.5% (5000 ppm). Although the color of our test fuel indicates a contaminant, the elemental analysis, and data from previous studies suggest the contamination is not significant enough to alter the lubricity of the test fuel. Both Wartsila and DUAP were contacted to ascertain the possibility of lubricant migration into the metering section of the pump. Neither company has responded to the requests for information. The injection pump bleed data indicates a fuel and oil mix in the injection pump at about 7 parts fuel to 1 part oil based on the calcium (Ca) results.

Both Wartsila and DUAP were contacted to ascertain the possibility of lubricant migration into the metering section of the pump. DUAP had not responded to the requests for information. Wartsila sent information pertaining to how the injection pump design is intended to separate fuel and oil. It is not entirely clear from the Wartsila text if the potential for lube oil contamination in the injected fuel is possible. However as noted, the level of contaminant from lube oil was very low, verified by lube oil element tracer, and previous work at SwRI indicated percentage levels of lubricant was needed in a fuel before lubricity changed.

Table 2. Elemental Analysis of USCG Fuel Injection Rig Samples

Property	Sample ID	376772	376774	376773	376941
	Description	FUEL TANK	INJECTOR EFFLUENT	INJECTION PUMP BLEED	CAMBOX OIL
ASTM D 5185 Elemental Analysis	Al	<1	<1	<1	1
	Sb	<1	<1	<1	<1
	Ba	<1	<1	<1	<1
	B	<1	<1	1	6
	Ca	<1	<1	223	1431
	Cr	<1	<1	<1	<1
	Cu	<1	<1	<1	8
	Fe	<1	<1	<1	6
	Pb	<1	<1	<1	2
	Mg	<1	1	69	457
	Mn	<1	<1	<1	<1
	Mo	<1	<1	<1	<1
	Ni	<1	<1	<1	<1
	P	<1	<1	164	1063
	Si	<1	<1	1	7
	Ag	<1	<1	<1	<1
	Na	<5	<5	<5	<5
	Sn	<1	<1	<1	<1
	Zn	<1	<1	175	1150
	K	<5	<5	<5	<5
Sr	<1	<1	<1	<1	
V	<1	<1	<1	<1	
Ti	<1	<1	<1	<1	
Cd	<1	<1	<1	<1	

Upon examination of the plunger to check activity it was noted a polished wear scar (Figure 16) was forming on the plunger. However, the wear scar development appears to miss the activated area of the plunger. The wear scar had not been noted prior to the JP-5 operation. However there is not any documentation to specifically indicate the scar was solely due to JP-5 operation. The location of activation was in an area indicated by DUAP as a high wear area. A possible cause for the deviation from that area is the drive system will not steadily hold pump speed above a 3/4-rack position. The wear scar appears to correspond to a position that represents 3/4-rack. Options for testing include running at 3/4-rack as with the other fuels to see if the poor lubricity fuel will affect the plunger wear regardless of the rack position. The other option was to allow greater speed variability and operate as close to full-rack as possible.

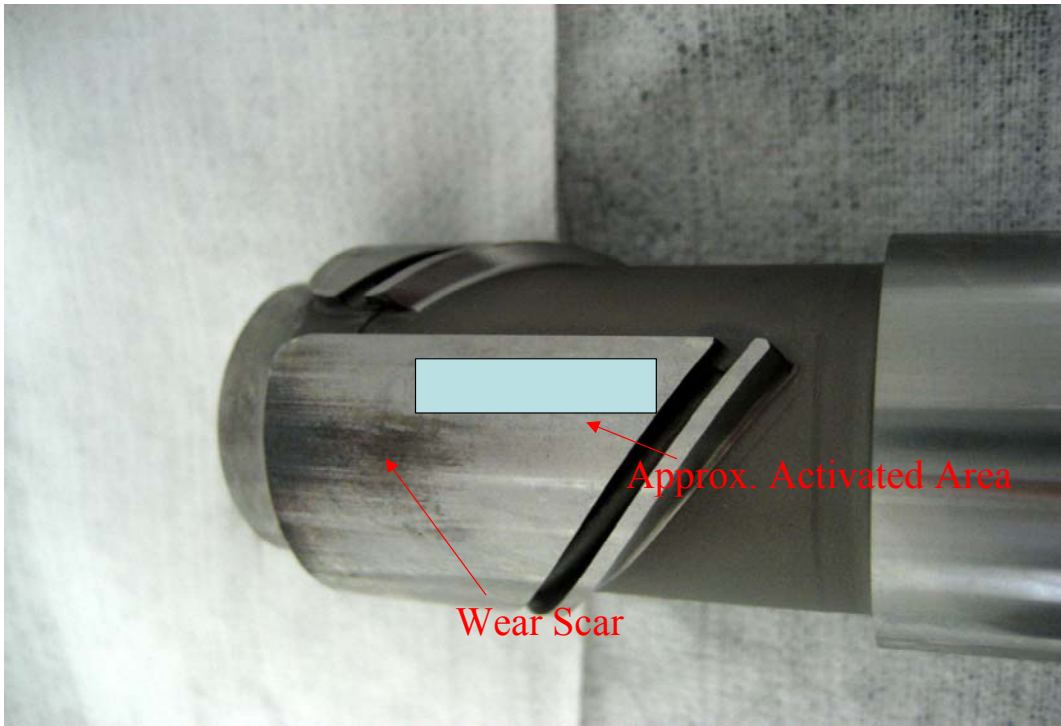


Figure 16. Developing Plunger Polished Wear Scar

The 3/4-rack position on the plunger corresponds to the location required to maintain a steady drive speed of 257-RPM with the hydraulic motor, flywheel combination. It was felt the rig could be operated at full-rack, however the drive speed would need to be increased to maintain a steady speed due to flywheel effect helping the drive motor. SwRI would relate the time at speed to injection events to the same number of injection events at the Healy injection pump operating speed. Because the USCG was interested in JP-5 effects, SwRI ran the 2000-g fuel with DCI-4A corrosion inhibitor at full-rack conditions to determine a wear rate.

The pump rig stand was operated at the full-rack position. Initially the drive would not exceed 210-RPM with the plunger set at full-rack. The rig was operating for about 2 hours at 210-RPM, with full-voltage to the hydraulic pilot valve, when suddenly the pump speed increased rapidly. After that point the injection pump could be run at 300-RPM and full rack with any speed variation. Possibly the pilot valve armature was hung up or the hydraulic pump swash plate was binding, but eventually speed control at all rack settings became possible. The pump was held at 257-RPM in order to perform an injection flow check at full-rack, to verify the injection pump was performing properly. At 257-RPM 3/4-rack the injection was typically 2200-ml/min, while

at full-rack the injection was 3000-ml/min. The injection pump was operated with 60-psi lubricant pressure and 100-120-psi fuel inlet pressure. The pump was operated overnight, for 23 hours. Two hours of operation were at 210-RPM and the remaining time at 300-RPM. This would be 26.2 hours equivalent operation at 257-RPM.

The plunger was pulled and an activity analysis indicated approximately 0.5-microns of material was removed from one side of the plunger. The other side showed activity levels in the statistical level of the readings. Inspection of the plunger indicated the wear scar had moved to the position desired. Unfortunately it appeared chips along the top edge of the plunger broke off and caused third body wear between the barrel and plunger, shown in Figure 17. The shiny edge indicated in Figure 17 is one location of a chip that flaked off. Figure 18 is picture of the mating barrel showing the effects of the third-body wear, and flakes or debris in the barrel. Inspection of the area shown in Figure 18 revealed the wear area to be pitted, like part of the barrel chipped out. Pitting is usually related to cavitation wear in fuel injection systems, and is usually seen in injection lines. SwRI investigated this wear between the barrel and plunger and documented the wear surfaces with Talysurf surface profilometer measurements to verify the surface conditions.



Figure 17. Sulzer Plunger with Wear Scars Formed from Third-Body Wear



Figure 18. Wear Scars in Injection Pump Barrel

8.0 DISCUSSION

The remaining project efforts attempted to establish whether the damage seen on the barrel and plunger were due to the surrogate JP-5 fuel or due to another factor. The barrel and plunger were sectioned and examined closely to try to ascertain the cause of failure.

Several photographs documenting unusual wear have been taken. Figure 19 is a photograph of one of two cracks found on the plunger helix. The crack is on the top edge that would travel across the pits that formed in the barrel. Evidence suggests that a ridge was forming on the lip of the plunger, and some of those lips may have chipped off. The ridge that appeared to form was not radial, but extended axially along the top edge of the plunger. The other crack found on the top of the plunger helix is shown in Figure 20. This area is around the idle position on the plunger. The area looks like a lip has formed and is in the process of chipping off.

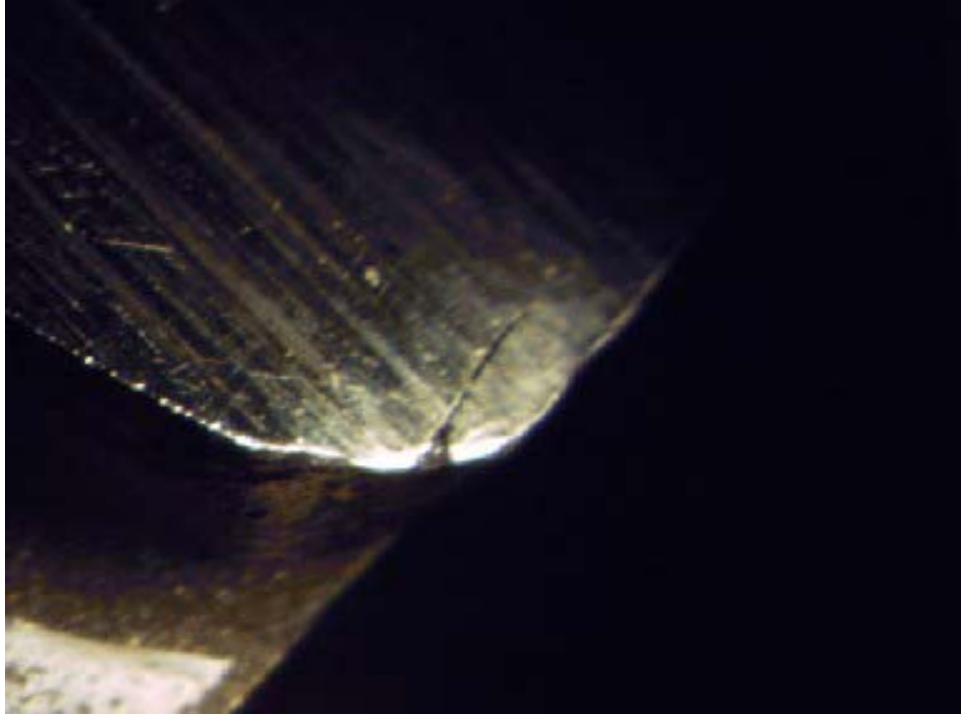


Figure 19. Sulzer Injection Pump Plunger Crack Formed at Edge of Helix

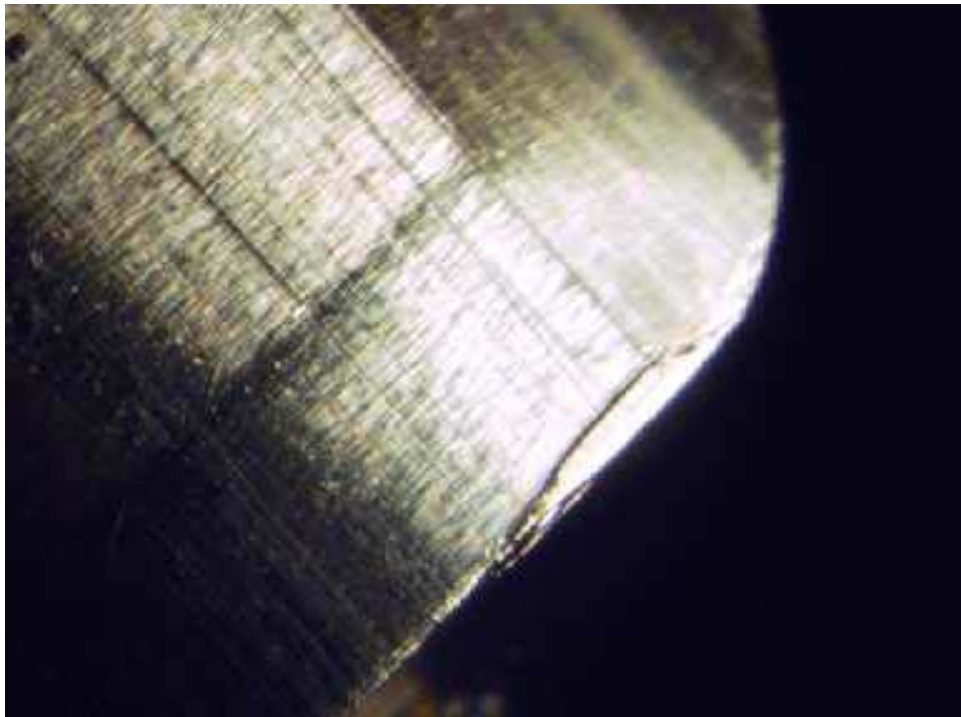


Figure 20. Second of Two Cracks Found on Sulzer Injection Pump Plunger Helix

SwRI looked for references that concern the conditions for cavitation in fuel injection barrel and plungers as they relate to the operating conditions, fuel viscosity, and fuel volatility of the surrogate JP-5 test. Cavitation in fuel injection equipment results in pitting due to surface fatigue in a localized area. The backlighted view of the same area in Figure 21 make the pits stand out due to the shadowing. Also seen in the backlighted view are some annular wear patterns that may correspond with machined areas on the plunger.

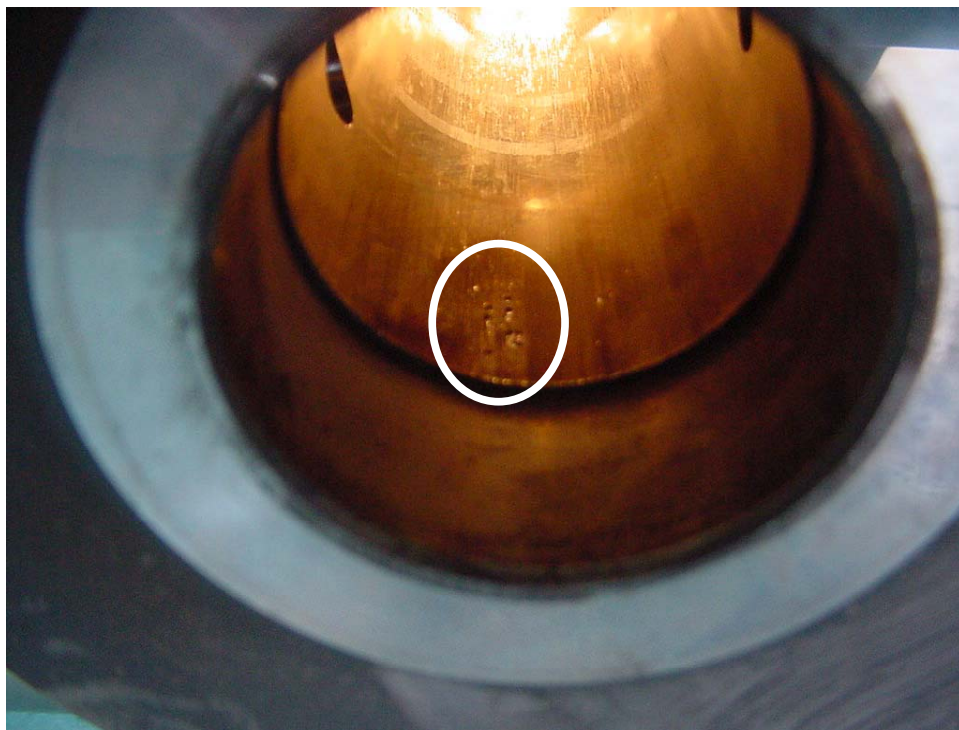


Figure 21. Sulzer Barrel with Shadows Revealing Pitting

The Talysurf surface profilometer trace shown in Figure 22 suggests there is bell-mouthed wear towards the top of the barrel. The wear is about 1.1-microns for a 18mm stroke from the bottom of the barrel on the left, towards the top of the barrel.

The Talysurf surface profilometer trace shown in Figure 23 is across one of the major pits of the injection pump barrel showing a scar depth of 47-microns. Of interest is the ploughing of

material by the plunger towards the top of the barrel. The ploughed material is likely what caused the severe wear on the plunger.

Figure 24 is a profilometer trace for a barrel that had the plunger activated improperly. This barrel only had 20 hours of break in operation on F3000 fuel. It is uncertain what the spike is because it is not visible to the eye. The trace indicates about 0.1-micron wear. At these small values the levelness of the component measured can show up as slight slope, but the trace appears to bell slightly towards the top of the stroke.

Taylor Hobson

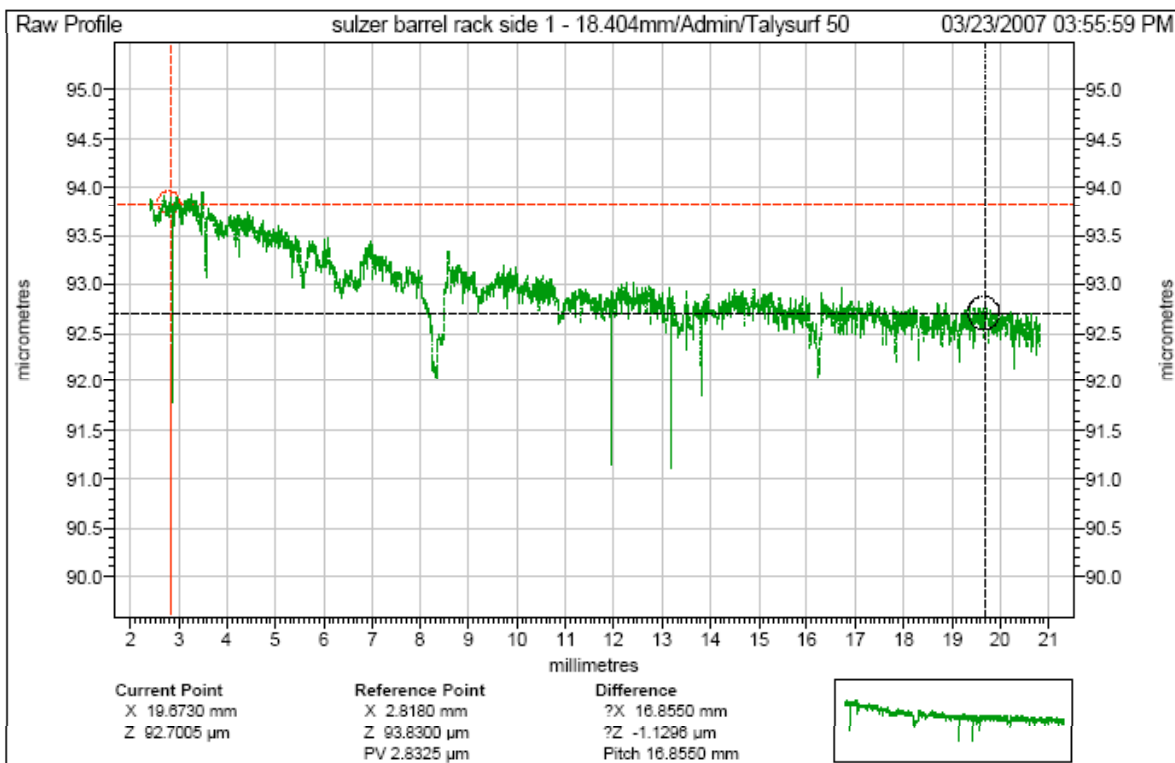


Figure 22. Surface Profilometer Trace of Barrel after being Leveled Showing Bell Mouth towards Top of Stroke

Taylor Hobson

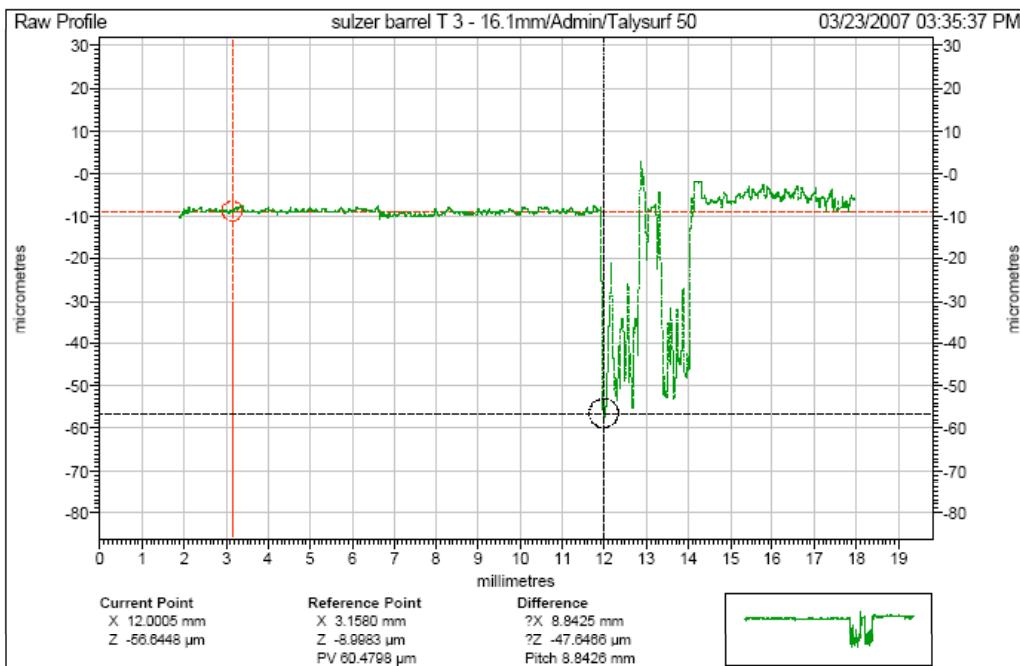


Figure 23. Surface Profilometer Trace across Pit from Injection Pump Barrel

Taylor Hobson

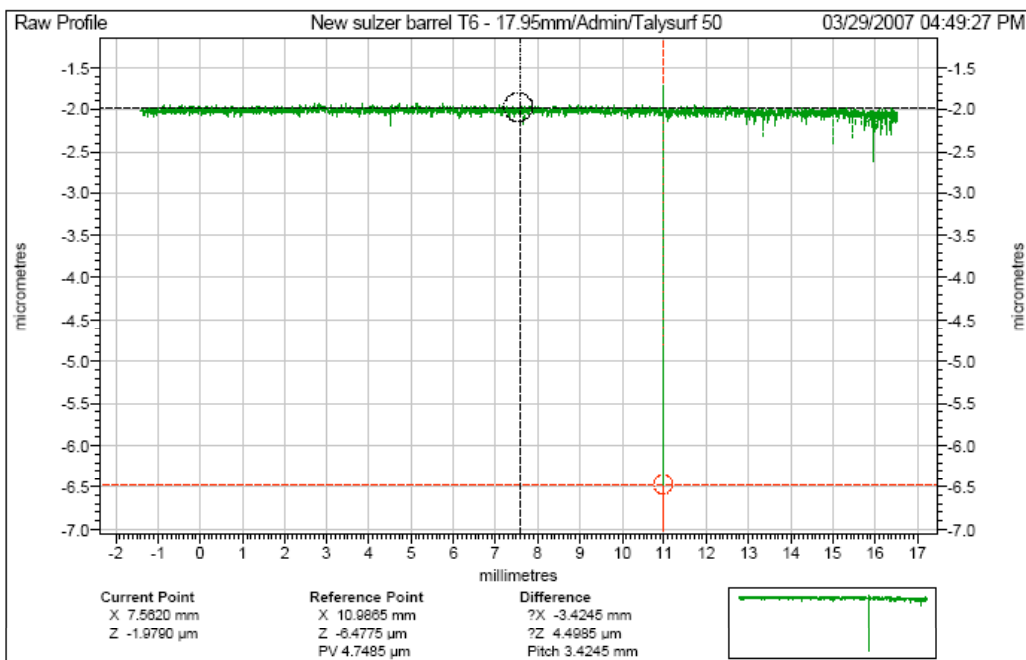


Figure 24. Surface Profilometer Measurement on a Low Time Barrel Operated on Diesel Fuel

The barrel and plunger were sectioned and examined in a Scanning Electron Microscope (SEM) to determine surface condition and elements in the worn areas. Areas examined included the base metal in an area without any wear, an area that appeared to undergo normal wear, and the scuffed area. Figure 25 shows the section barrel, with the scored area that was examined.

Figure 26 shows a close up of the damaged area of the section barrel. The Secondary Electron Detector (SED) image in Figure 27 is of the wear and scored material seen in the upper -mid section of Figure 26, and was taken with a Scanning Electron Microscope. The image shows how the material appears to be fracturing, as if being ploughed by the plunger. The back-scatter image of Figure 28 highlights the fracture lines in this worn area of the barrel. Another scored area of the barrel is shown in a back-scatter image (Figure 29), which also shows the fracture lines from plowing between the barrel and plunger. These results suggest a third body between the barrel and plunger initiated the wear.

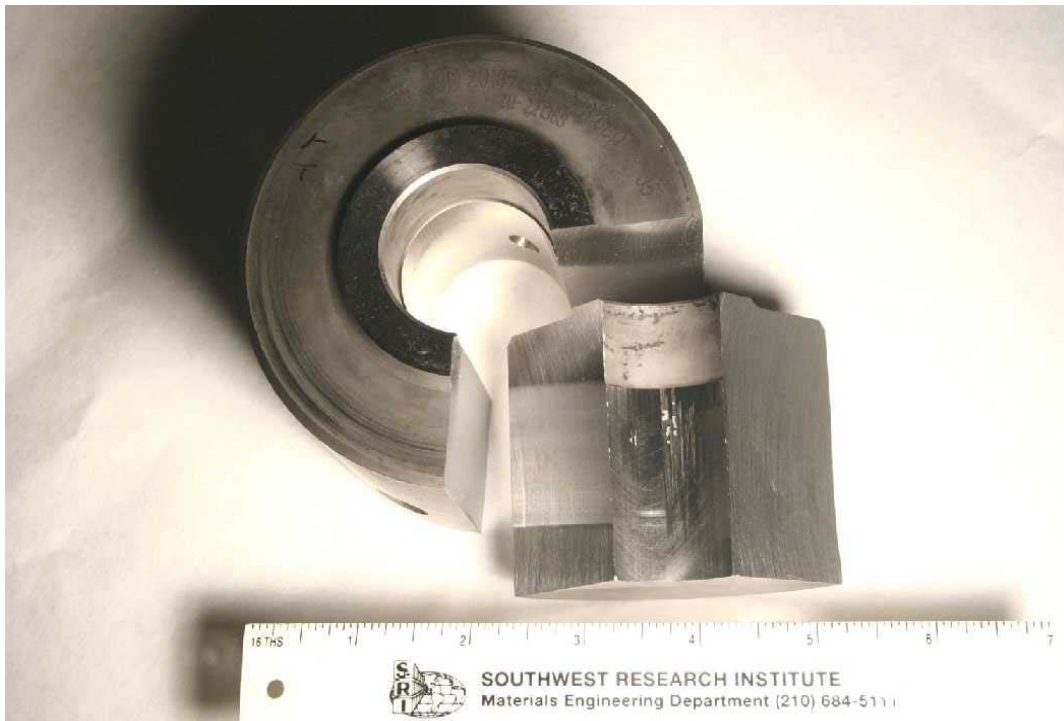


Figure 25. Sectioned Barrel with Scuffed Area Examined

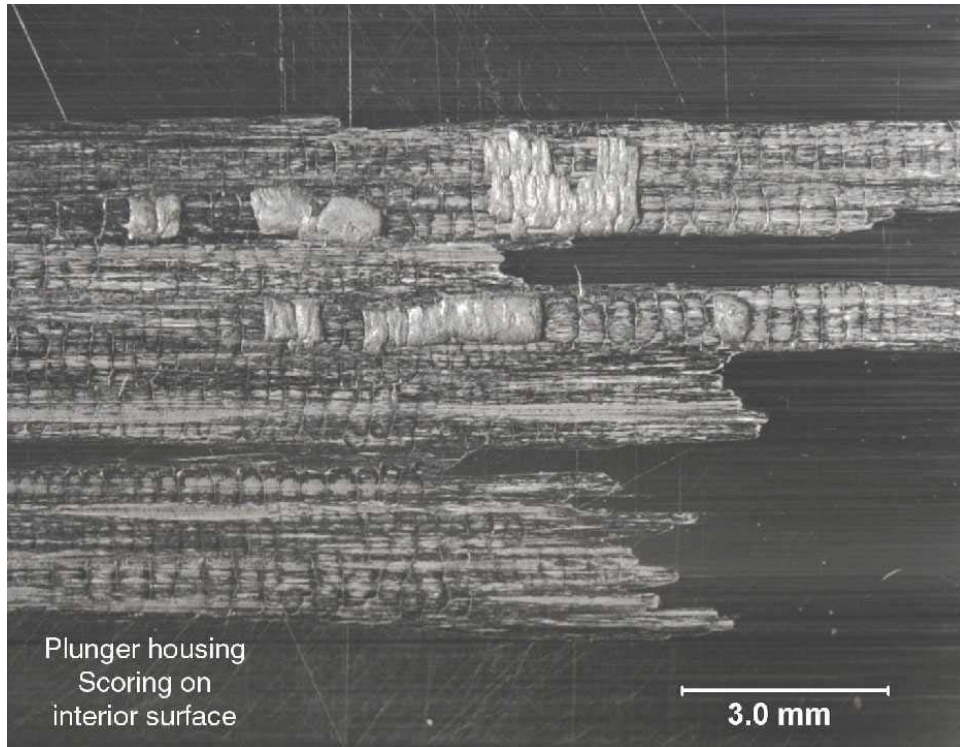


Figure 26. Scuffed and Worn Section of Barrel

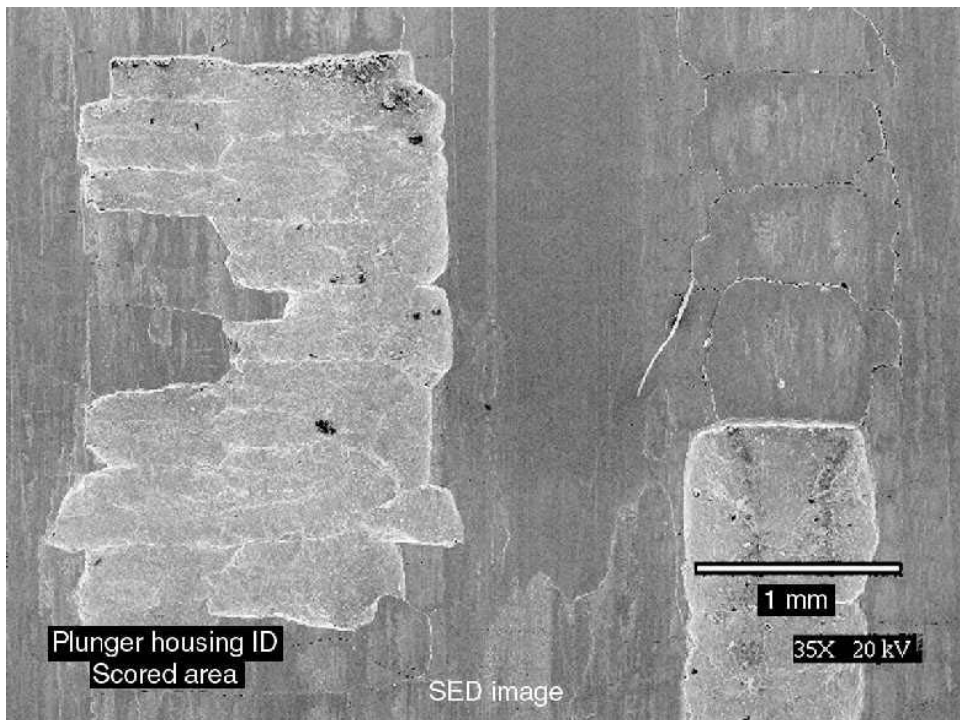


Figure 27. Scanning Electron Microscope Image of Scuffed Area of Barrel

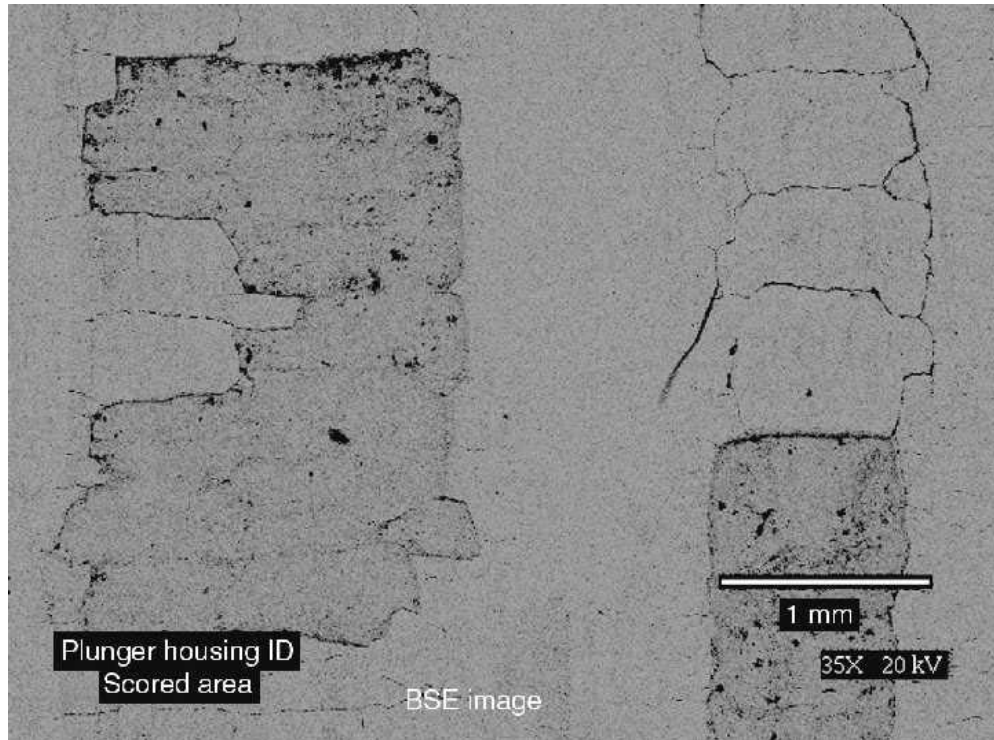


Figure 28. Back Scatter Scanning Electron Image of Scored Barrel

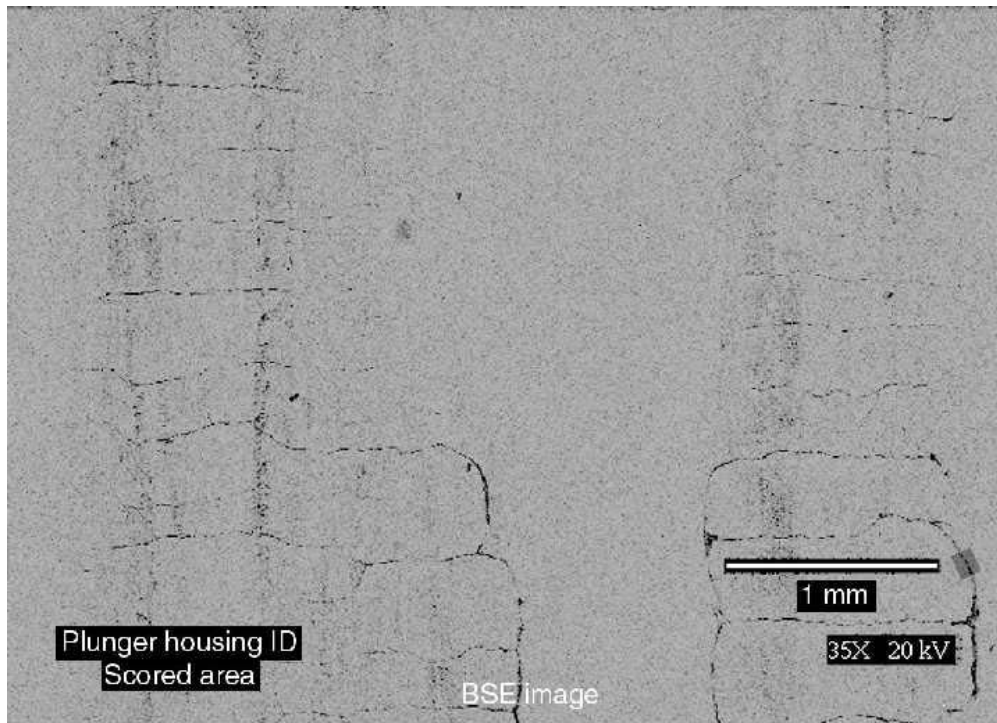


Figure 29. Back Scatter Image of a Different Scored Area of the Barrel

An elemental analysis was performed on an area of the barrel that did not reveal wear and on one of the scored areas. The un-scored area elemental results matched closely with the base metal elemental results. Figure 30 details the elemental analysis of the scored area and Figure 31 details the elements found in a scored area of the barrel. An element that shows up in the scored area is aluminum, which does not show up in the base metal or un-scored area elemental scans. The quantity of aluminum is small, thus alloying elements of are not seen so it is not certain if the aluminum is from an alloy or from an abrasive such as aluminum oxide.

The heat treatment case depth was checked for the barrel. Figure 32 shows the etched barrel section with the depth of the case hardening around 300-microns.

Plunger housing
 Interior surface
 Unscored area

Chi-sqd = 3.11 Livetime = 300.0 Sec.
 Standardless Analysis

PROZA Correction Acc.Volt.= 20 kV Take-off Angle=43.06 deg Tilt = 1 deg
 Number of Iterations = 3

Element	k-ratio (calc.)	ZAF	Atom % Wt %	Element Wt %	Err. (1-Sigma)
Si-K	0.0055	1.787	1.93	0.98	+/- 0.05
V-K	0.0112	0.895	1.09	1.00	+/- 0.08
Cr-K	0.0663	0.812	5.74	5.38	+/- 0.10
Mn-K	0.0034	1.025	0.35	0.34	+/- 0.06
Fe-K	0.8956	1.011	89.87	90.53	+/- 0.34
Mo-L	0.0132	1.340	1.02	1.76	+/- 0.09
Total			100.00	100.00	

Note: Results do not include elements with Z<11 (Na)

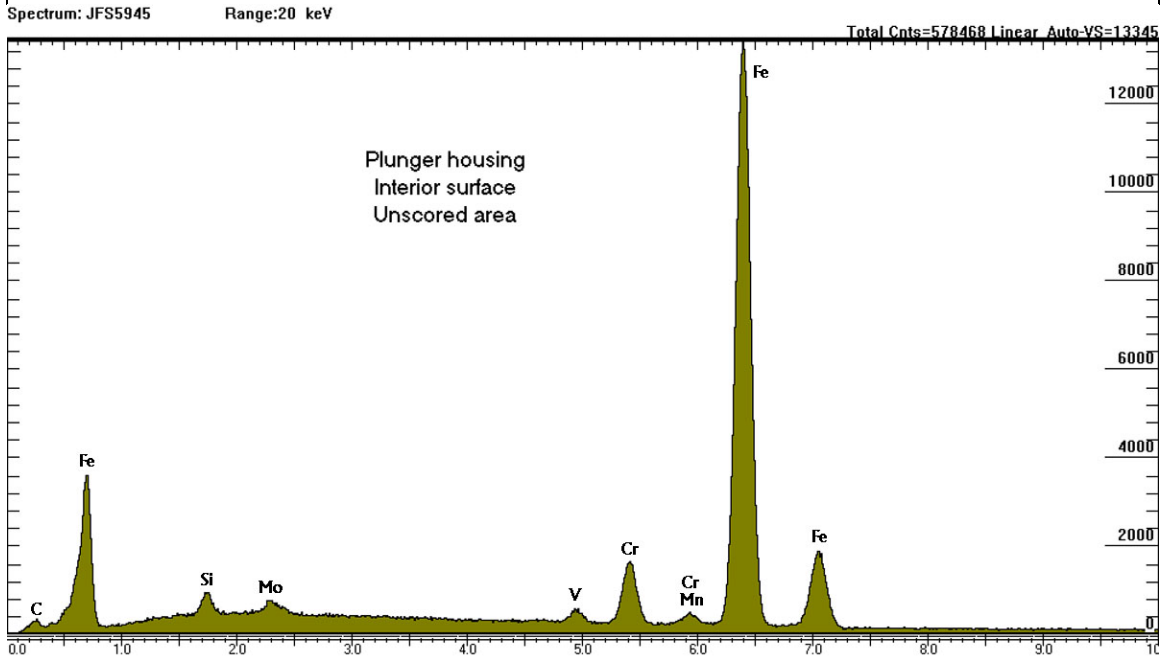


Figure 30. Elemental Scan Results for Unscored Area of Barrel

Plunger housing
 On score
 Area 2

Chi-sqd = 5.19 Livetime = 515.0 Sec.
 Standardless Analysis

PROZA Correction Acc.Volt.= 20 kV Take-off Angle=43.06 deg Tilt = 1 deg
 Number of Iterations = 3

Element	k-ratio (calc.)	ZAF	Atom % Wt %	Element Wt %	Err. (1-Sigma)
Al-K	0.0009	2.454	0.45	0.22	+/- 0.03
Si-K	0.0040	1.797	1.41	0.71	+/- 0.04
V -K	0.0078	0.888	0.75	0.69	+/- 0.06
Cr-K	0.0549	0.803	4.70	4.41	+/- 0.07
Mn-K	0.0068	1.025	0.71	0.70	+/- 0.09
Fe-K	0.9114	1.009	91.22	91.94	+/- 0.27
Mo-L	0.0099	1.342	0.77	1.33	+/- 0.11
Total			100.00	100.00	

Note: Results do not include elements with Z<11 (Na)

Spectrum: JFS5944

Range:20 keV

Total Cnts=1058364 Linear Auto-VS=24640

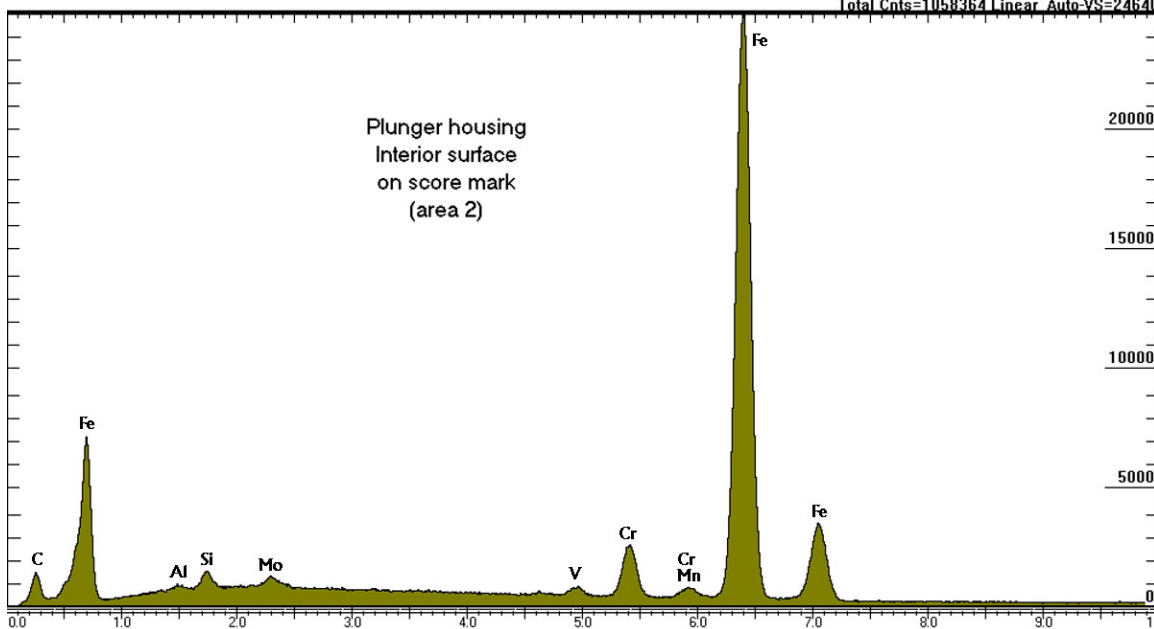


Figure 31. Elemental Scan Results for Scored Area of Barrel



Figure 32. Barrel Etched to Determine Case Hardness Depth

An image of a fuel fill/spill port on the fuel injection barrel is shown in Figure 33, with evidence of chipping on the upper edge of the port. Chipping on the port has been known to occur with debris that is sheared between the plunger and port edge during the injection process. The image suggests some debris may have initiated the chipping of the port edge, than the port edge debris caused the further damage to the barrel and plunger. However the damage around the port may be due to erosion when the high-pressure fuel exits through the port during the spill event. This is further supported by the fact that the barrel is not damaged immediately above and below the ports.

The damaged plunger was also examined. Figure 34 shows a damaged section of the plunger that was examined. The SED image of a worn area of the plunger is shown in Figure 35. Evident in the figure is vertical and circumferential fracture lines. Figure 36 also shows circumferential fracture lines evident on the plunger surface as highlighted by back scatter imaging. A magnified image of the cracking is highlighted by backscatter imaging in Figure 37. On the cracking, plastic deformation of the hardened surface layer appears to be causing cracks that propagate parallel to the surface and cause small pieces to eventually spall off. The depth of the carburized case is around 300 micron as shown in Figure 38. An undamaged area of the plunger is shown in Figure 39. The core and surface hardness of the plunger were found to be Rc 34 for the plunger core hardness and Rc 56 for the surface hardness.

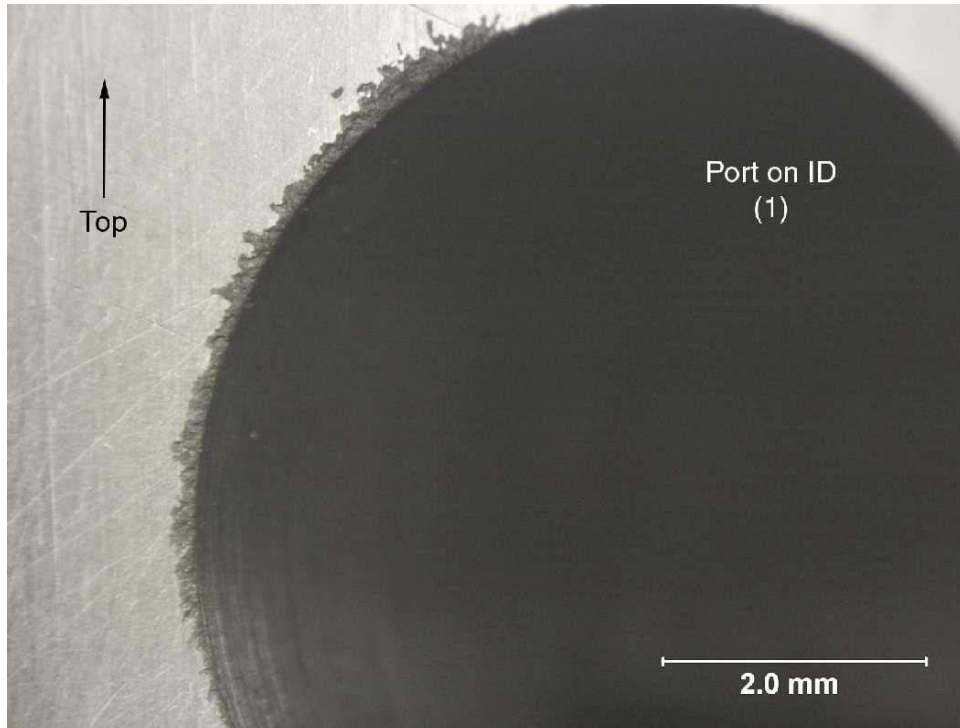


Figure 33. Barrel Fuel Port with Evidence of Chipping on Port Edge

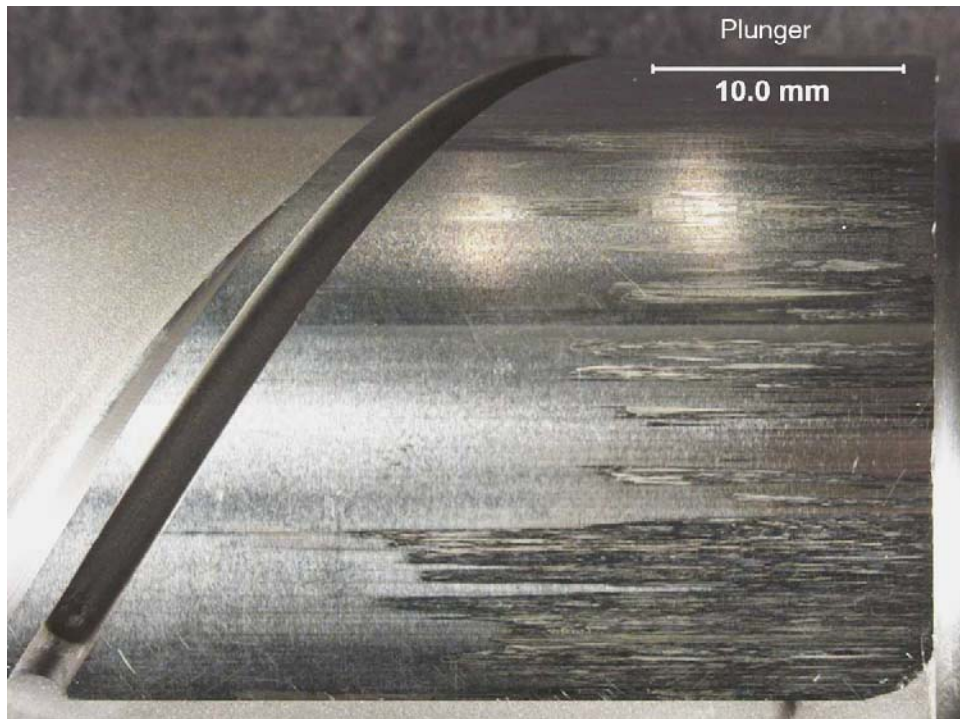


Figure 34. View of Damaged Injection Pump Plunger

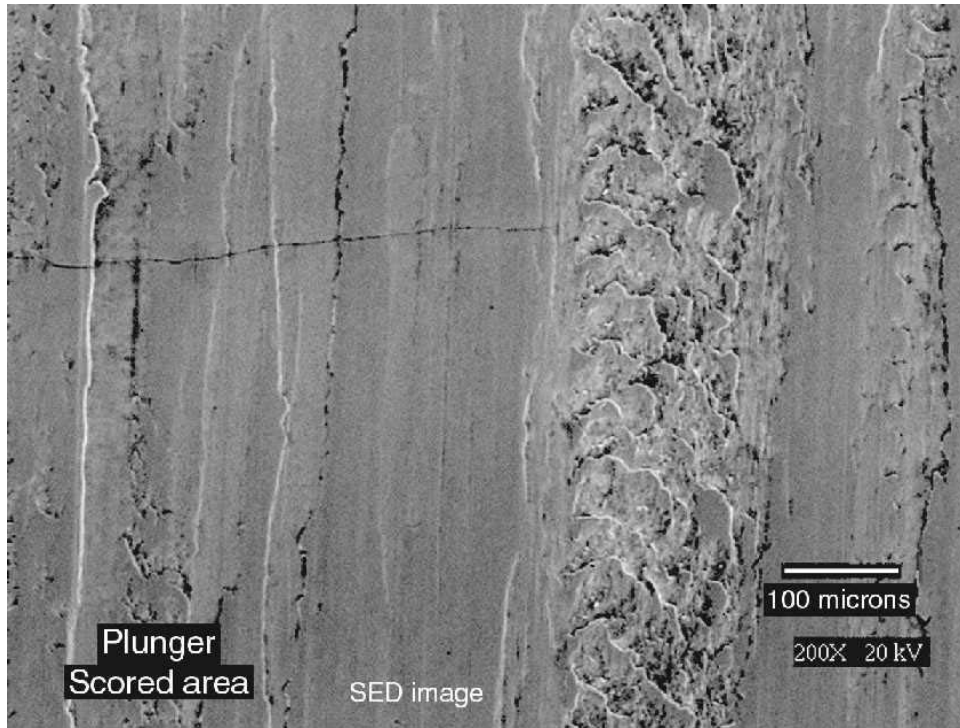


Figure 35. Plunger Scored Area with Evident Fracture Lines

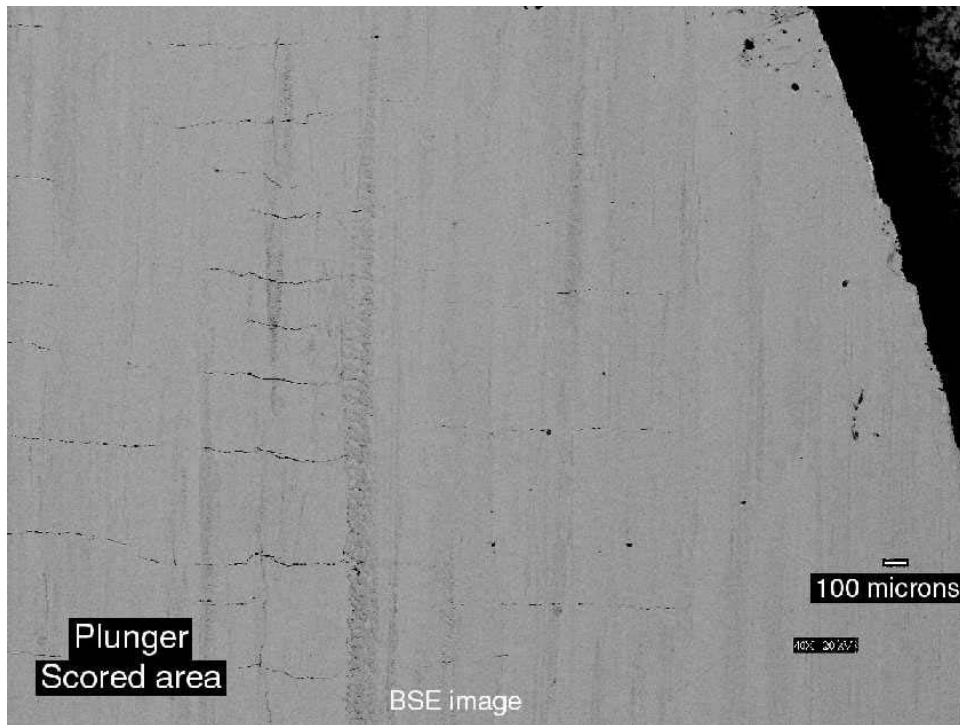


Figure 36. Circumferential Fracture Lines on Plunger Surface

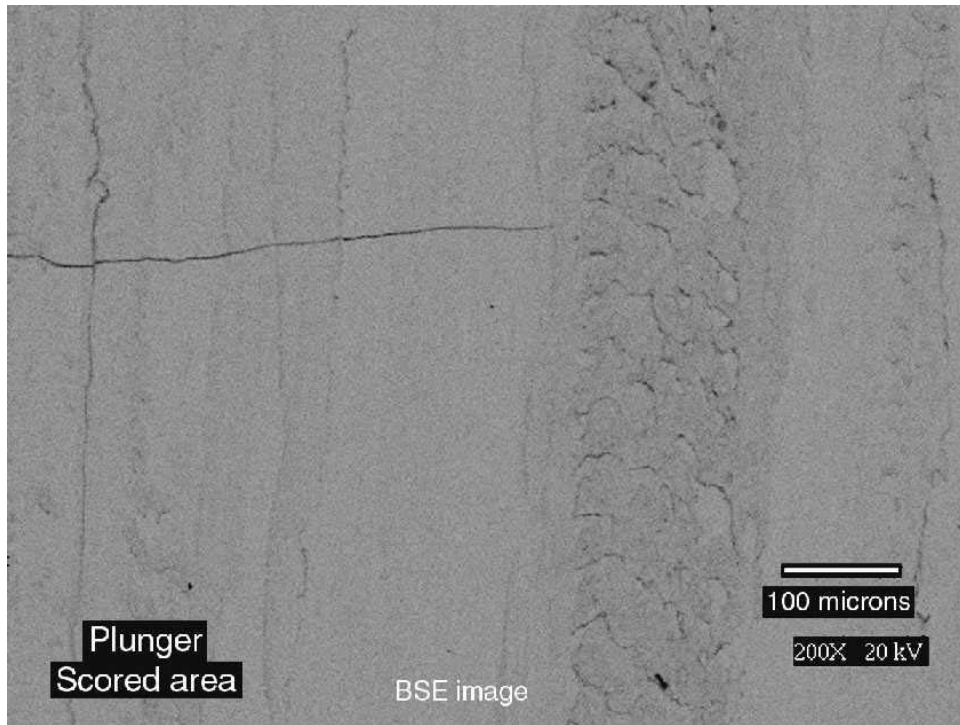


Figure 37. Close up of Cracked Surface of Plunger

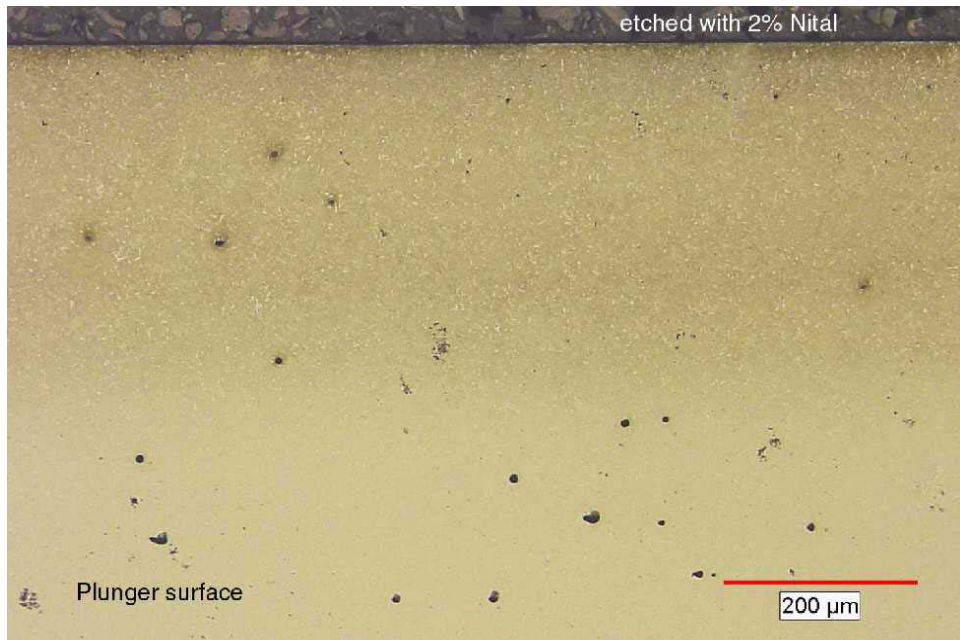


Figure 38. Case Depth of the Fuel Injection Plunger

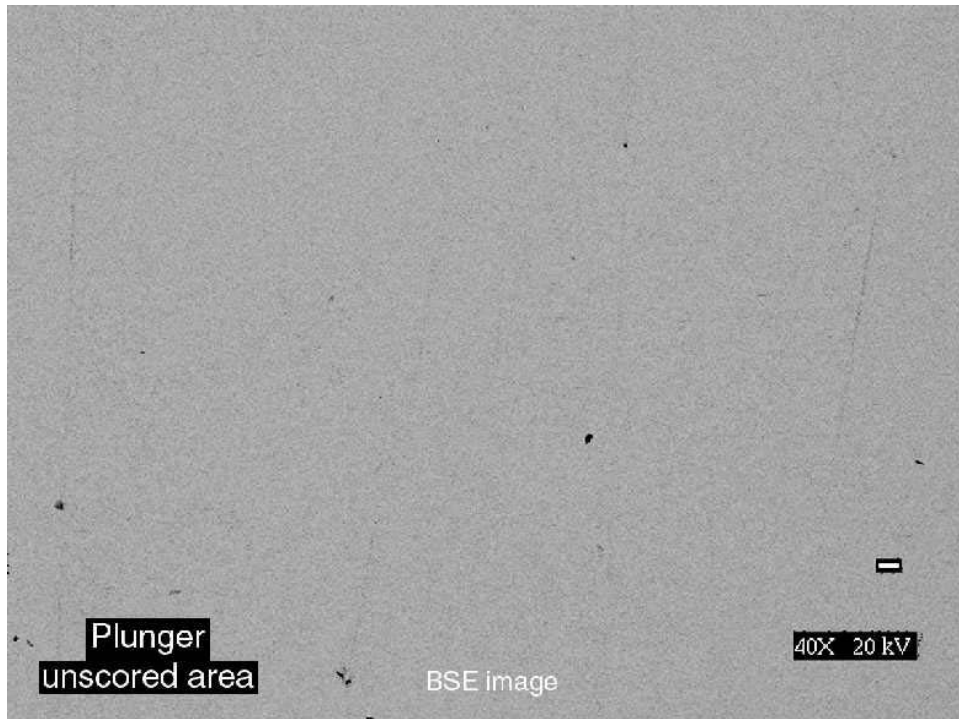


Figure 39. View of an Undamaged Area of the Injection Pump Plunger

Elemental scans were performed on the base material and on a damaged section of the plunger to determine if foreign materials were present in the wear scars. As seen with the barrel, the plunger revealed a small aluminum peak in the damaged areas as shown in Figure 40. The undamaged area did not show an aluminum peak as seen in Figure 41.

The system fuel filter was washed with a solvent, then the solvent was filtered through a Millipore filter, then the filter was dried. The remaining debris was analyzed for elemental composition and the results are shown in Figure 42. Particles of iron and aluminum alloy were found in the filter debris as shown in Figure 43. The aluminum may have come from the threads of the filter element holder when it was tightened after the filter was changed. The iron could be wear debris that is recirculating through the fuel injection system. It is known that particulate can be pumped through filter media due to pressure pulsations. The pressure pulsations for the single pumping element system were quite severe.

The elemental analysis suggests either aluminum or iron particles were pumped through filter and resulted in damage to the fuel injection pump. There is very little evidence of scoring directly around the fill/spill ports that suggests the damage was not due to particle shearing between the port and the plunger. The damaged areas show evidence of aluminum, as does the fuel filter debris.

The fuel injector showed evidence of wear also. Figure 44 shows the fuel injector pintle with a developed wear scar. It is likely the wear scar developed due to material from the barrel and plunger migrating downstream with the fuel.

Plunger
 On score mark
 Area 3

Chi-sqd = 3.55 Livetime = 300.0 Sec.
 Standardless Analysis

PROZA Correction Acc.Volt.= 20 kV Take-off Angle=35.63 deg Tilt = 1 deg
 Number of Iterations = 3

Element	k-ratio (calc.)	ZAF	Atom % Wt % (1-Sigma)	Element	Wt %	Err.
Al-K	0.0005	2.751	0.31	0.15	+/- 0.03	
Si-K	0.0031	1.971	1.20	0.61	+/- 0.05	
Ca-K	0.0009	0.955	0.11	0.08	+/- 0.02	
V -K	0.0055	0.893	0.54	0.49	+/- 0.06	
Cr-K	0.0438	0.807	3.77	3.53	+/- 0.08	
Mn-K	0.0054	1.026	0.56	0.55	+/- 0.10	
Fe-K	0.9263	1.008	92.83	93.41	+/- 0.29	
Mo-L	0.0085	1.379	0.68	1.17	+/- 0.12	
Total			100.00	100.00		

Note: Results do not include elements with Z<11 (Na)

Spectrum: JFS5937

Range:20 keV

Total Cnts=883223 Linear_Auto-VS=22032

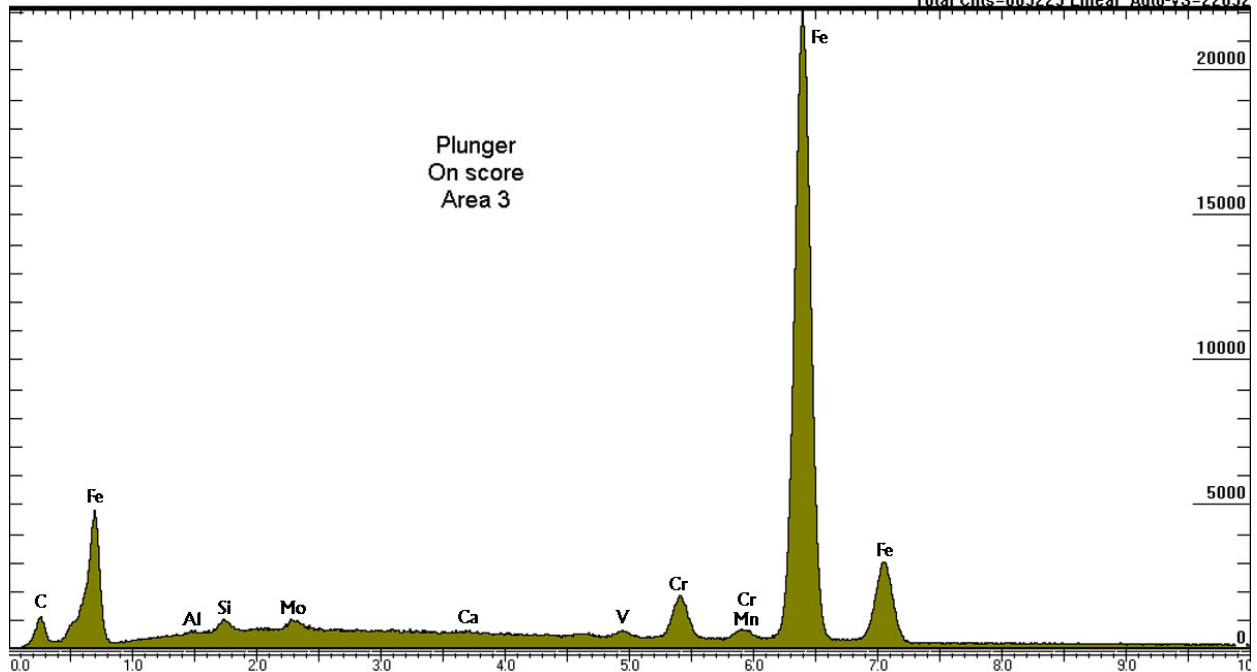


Figure 40. Elemental Scan of Plunger Damaged Surface

Plunger
Base metal

Chi-sqd = 3.31 Livetime = 300.0 Sec.
Standardless Analysis

PROZA Correction Acc.Volt.= 20 kV Take-off Angle=40.85 deg
Number of Iterations = 3

Element	k-ratio (calc.)	ZAF	Atom %	Element Wt % (1-Sigma)	Wt % Err.
Si-K	0.0027	1.849	1.00	0.51	+/- 0.04
V-K	0.0037	0.880	0.35	0.32	+/- 0.06
Cr-K	0.0394	0.792	3.34	3.12	+/- 0.08
Mn-K	0.0078	1.024	0.81	0.80	+/- 0.11
Fe-K	0.9385	1.006	94.02	94.42	+/- 0.30
Mo-L	0.0061	1.353	0.48	0.83	+/- 0.12
Total			100.00	100.00	

Note: Results do not include elements with Z<11 (Na)

Spectrum: JFS5946

Range:20 keV

Total Cnts=808251 Linear Auto-YS=19632

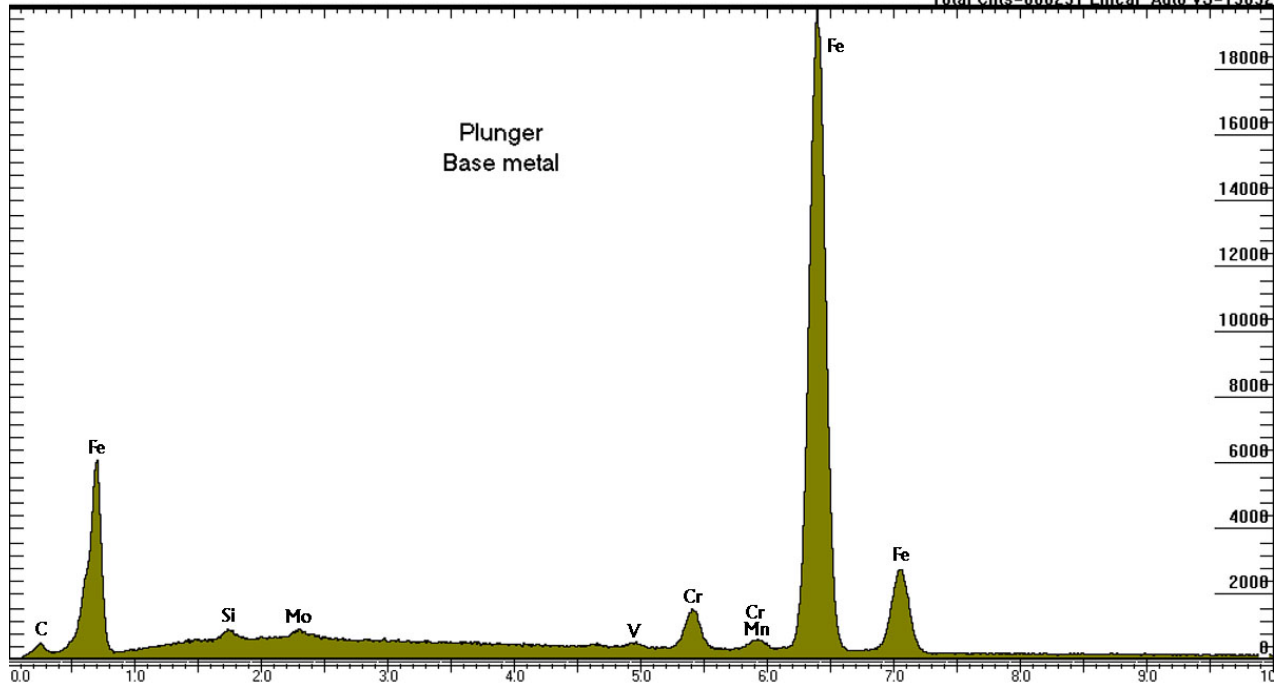


Figure 41. Elemental Scan of Plunger Base Material

Fuel filter debris
from USCG injector system

Chi-sqd = 24.14 Livetime = 300.0 Sec.
Standardless Analysis

PROZA Correction Acc.Volt.= 20 kV Take-off Angle=46.25 deg Tilt = 2 deg
Number of Iterations = 5

Element	k-ratio	ZAF	Atom % (calc.)	Element Wt %	Wt % (1-Sigma)	Err.
Na-K	0.0052	3.467	3.08	1.81	+/- 0.18	
Mg-K	0.0096	2.426	3.75	2.33	+/- 0.26	
Al-K	0.0999	1.912	27.77	19.11	+/- 0.27	
Si-K	0.0578	1.786	14.41	10.32	+/- 0.19	
P -K	0.0038	1.651	0.79	0.63	+/- 0.05	
S -K	0.0154	1.375	2.59	2.12	+/- 0.11	
Cl-K	0.0083	1.294	1.19	1.07	+/- 0.10	
K -K	0.0070	1.105	0.78	0.78	+/- 0.05	
Ca-K	0.0859	1.041	8.74	8.94	+/- 0.14	
Ti-K	0.0187	1.086	1.66	2.02	+/- 0.12	
Cr-K	0.0087	1.005	0.66	0.88	+/- 0.07	
Mn-K	0.0040	1.105	0.32	0.44	+/- 0.09	
Fe-K	0.4062	1.076	30.70	43.73	+/- 0.34	
Ni-K	0.0031	1.104	0.23	0.35	+/- 0.11	
Cu-K	0.0122	1.154	0.87	1.41	+/- 0.15	
Zn-K	0.0357	1.144	2.45	4.08	+/- 0.20	
Total			100.00	100.00		

Note: Results do not include elements with Z<11 (Na)

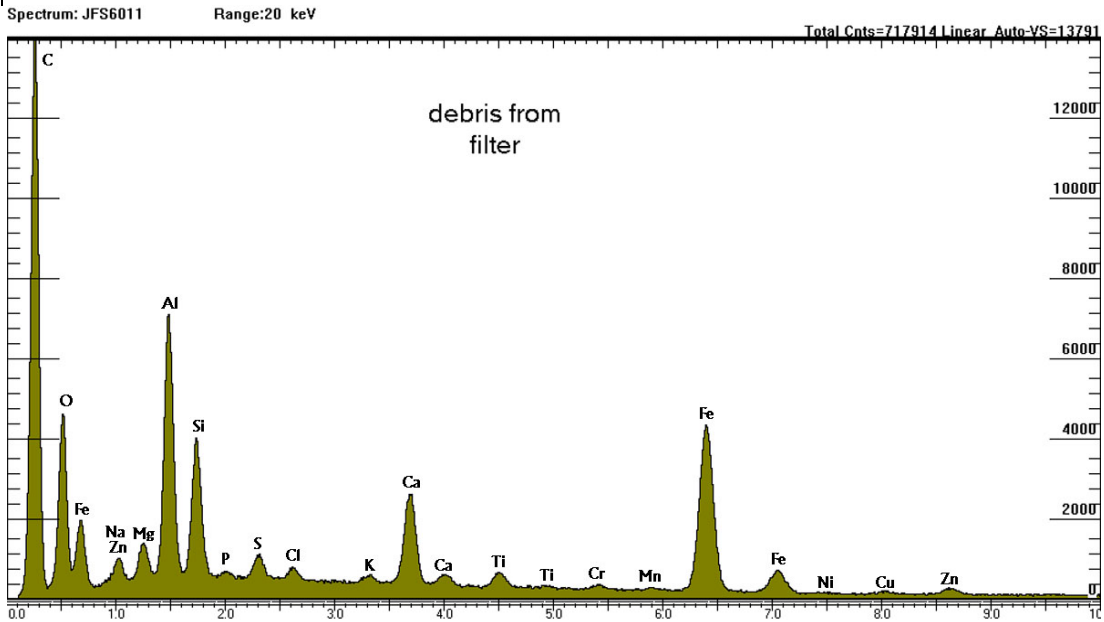


Figure 42. Fuel Filter Debris Elemental Analysis

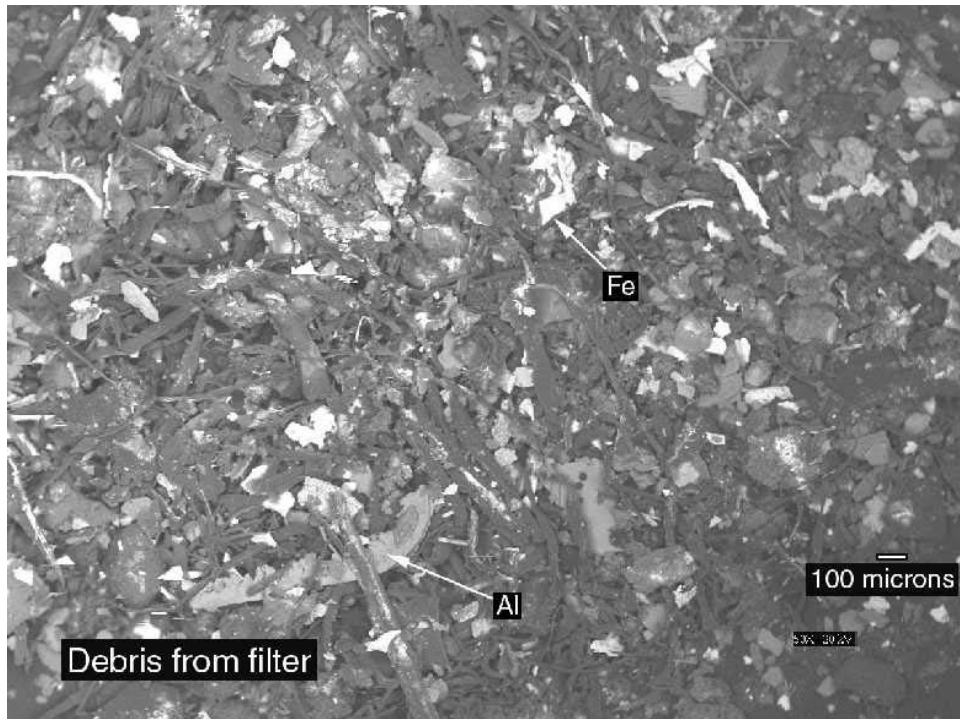


Figure 43. Aluminum and Iron Particles from Fuel Filter Debris



Figure 44. Wear Scar on Fuel Injector Pintle

The analysis suggests it is likely a contaminant entered the fuel injection system and caused the rapid wear. It is likely the contaminant may have been introduced during the disassembly/assembly processes that were carried out in order to check plunger activity. Although considerable care was taken when handling the components, it is possible the plunger could have been damaged in the disassembly/assembly processes.

However the depth of the wear scars and evidence of pitting suggests there may have been other mechanisms in effect. The engine and fuel injection equipment manufacturers were queried to determine if cavitation could have been a root cause of the failure; the manufacturers were asked the following questions:

- An item that is not understood is whether cavitation in the fuel injection pump could have resulted in surface fatigue that resulted in the damage seen between the barrel and plunger?
- Could operating the system with a lower viscosity fuel at an elevated operating speed have induced and accelerated cavitation?
- Can cavitation result in the damage seen in less than 24 hours of operation?

The engine and fuel injection equipment manufacturers declined to respond to the above questions concerning the experiment and the wear observed in the study.

9.0 CONCLUSIONS

The following conclusions could be made from this work:

- Radioactive Tracer methods can be sensitive, but with heavy metal objects shielding the radioactive source, in-situ measurements can be imprecise.

- The Radioactive Tracer method utilized for the 12ZAV40S plunger was not sensitive enough to determine wear rates between different lubricity fuels within 72 hours of operation of a high durability fuel system.
- Within a resolution of 0.14-microns, there was not a difference in wear between a 3000-g and a 2000-g scuffing load fuel based on radioactive tracer measurements.
- Precise knowledge of wear locations are critical in order to determine fuel wear rates with Surface Layer Activation methods.
- Rapid wear and damage can occur if contaminants, improper handling, or other factors such as cavitation are introduced.
- Damage may have been caused by the expanded operating speed of the test fuel injection system, which was beyond the speed range of the engine in service.

10.0 SUMMARY AND RECOMMENDATIONS

The use of aviation turbine fuel, with corresponding lower viscosity and fuel lubricity, in large bore diesel engines has been investigated over the years (11, 12). Typically the large bore engines burning the aviation turbine fuel were at remote locations that supported aviation activities, or were in colder climates where lower viscosity fuel was required. Generally the aviation turbine fuel use was acceptable in performance and in a location using Jet-A, acceptable with a reduction in component durability (13).

Recent studies of aviation turbine fuel grade JP-5 use in Naval shipboard engines have also indicated the use to be satisfactory (14, 15). However the JP-5 use in the recent surveys has only been in large engines that serve as Ship Service Diesel Generators and Emergency Diesel Generators. There is not a large body of information on Main Propulsion use medium-speed diesel engines burning JP-5. The recent surveys do suggest that since the engines are qualified

MILSPEC engines, they would have been required to function on JP-5 as and alternative or emergency fuel.

The four Sulzer 12ZAV40S engines of the USCG Healy serve as main propulsion and ship service power. The icebreaking, rescue, research, and polar duty of the USCG Healy suggest an understanding of the JP-5 effects on the fuel injection system components is critical. As such the results generated in this study are inconclusive. Operation with the radioactive tracer did suggest that the wear rates were below the sensitivity of the method. Essentially the fuel injection pump had operated approximately 150 hours on two different fuels and did not show any reduction in radioactivity of the test component after correcting for the isotope half-life. The sensitivity of the SLA method was 0.14-microns of wear in the activated area. The activation profile was based on data shown in Table 3 from a technical manual. The allowable wear clearance of the plunger over the 24000-hour expected fuel injection pump life indicated in the technical manual, suggested the depth profile target for obtaining wear measurements in approximately 40 hours of operation.

Table 3. Sulzer 12ZAV40S Fuel Injection Pump Clearance and Wear

**New
Sulzer
Diesel
12ZAV40S**

**Clearance and Wear Table
Fuel Injection Pump**

Maintenance

0352-1/A1

Group No.	Key No.	Part Description (1 mm/25.4mm/in. =0.0394 inches)	Direction of Measuring	Nominal Dimension inches	Nominal Dimension mm	Maximum Clearance (due to wear)	
						inches	mm
Fuel Injection Pump							
5501	1	Pump Plunger	O.D.	1.3790	35	0.009	0.230
		Plunger guide	I.D.	1.3790	35		
		Clearance	radial	0.0006 to 0.0006	0.014 to 0.016		
5501	2	Control rack	O.D.	1.1820	30	0.0059	0.150
		Housing (bore)	I.D.	1.1820	30		
		Clearance	radial	0.0016 to 0.0037	0.04 to 0.094		
5501	3	Guide piston	O.D.	5.1220	130	0.0059	0.150
		Pump housing	I.D.	5.1220	130		
		Clearance	radial	0.0017 to 0.0048	0.043 to 0.123		
5501	4	Bearing bushing	O.D.	2.3640	60	0.0039	0.100
		Roller	I.D.	2.3640	60		
		Clearance	radial	0.0012 to 0.0035	0.03 to 0.090		
5501	5	Roller pin	O.D.	1.9700	50.00	0.0039	0.100
		Bearing bushing	I.D.	1.9700	50.00		
		Clearance	radial	0.0010 to 0.0032	0.025 to 0.080		
5501	6	Roller pin	O.D.	1.9700	50.00	0.0032	0.080
		Guide piston (bore)	I.D.	1.9700	50.00		
		Clearance	radial	0.0004 to 0.0020	0.009 to 0.050		
5501	7	Axial clearance, bearing bushing	radial	0.0079 to 0.0158	0.20 to 0.400	0.0197	0.500
5501	7a	Total clearance between guide piston and roller with locating discs	radial	0.0102 to 0.0213	0.26 to 0.540	0.0276	0.700

Parts conditions did indicate however that limitations of the drive system meant that wear was occurring at a location other than the area activated. When the pump was operated at a condition of full-rack, and wear did occur at the activated location, the wear observed was due to scuffing on one side of the plunger. The diametrically opposite side of the plunger, which was also activated, did not show any activity loss after correcting for half-life. This result suggests the

scuffing wear of the plunger was due to an effect other than the lubricity of the test fuel. These confounding effects could be cavitation or contamination.

Due to the issues involved with in-situ measurements, testing with off-line measurements were performed that required disassembly of the fuel injection pump to remove the barrel and plunger. In service the fuel injection pump would certainly never be disassembled. During the evaluations the fuel injection pump was disassembled several times, with each time there being a risk of contamination being introduced into the fuel injection pump. Another possible risk was nicking the plunger during handling or assembly that could introduce a burr into the fuel injection pump. When operating on JP-5 fuel, the fuel injection pump drain was discharging 10.2 liters of fuel every four hours. To keep the recirculating fuel system sump at an appropriate level, makeup fuel was put in every four hours operation. Each batch of makeup fuel could have been a risk factor for the introduction of contaminants. Although there was a fuel filter in the system, it has been documented that particles can be pumped through a fuel filter due to hydraulic action and flexing of the membrane due to pressure fluctuations. For the 12ZAV40S test stand the pressure fluctuations were quite severe. A long-term test with a single pass fuel system, where the components are documented at the beginning and end would eliminate the risk of contamination due to handling and fuel additions.

Cavitation erosion is a failure mechanism in diesel fuel injection systems. Studies have shown that cavitation in fuel injection pumps can reduce durability and in a pump-line system can show up around the ports of the barrel and plunger assembly (16). Cavitation is due to the collapse of fuel vapor bubbles that can lead to high-pressure spikes that may result in surface fatigue or pitting. Some modeling efforts suggest the venting from the high-pressure chamber above the plunger, either across the seal path, or when metering port cut-off or spill port opening occurs, can cause bubble clouds to form (17). The collapse of these bubble clouds can generate localized high pressures in a fuel injection pump.

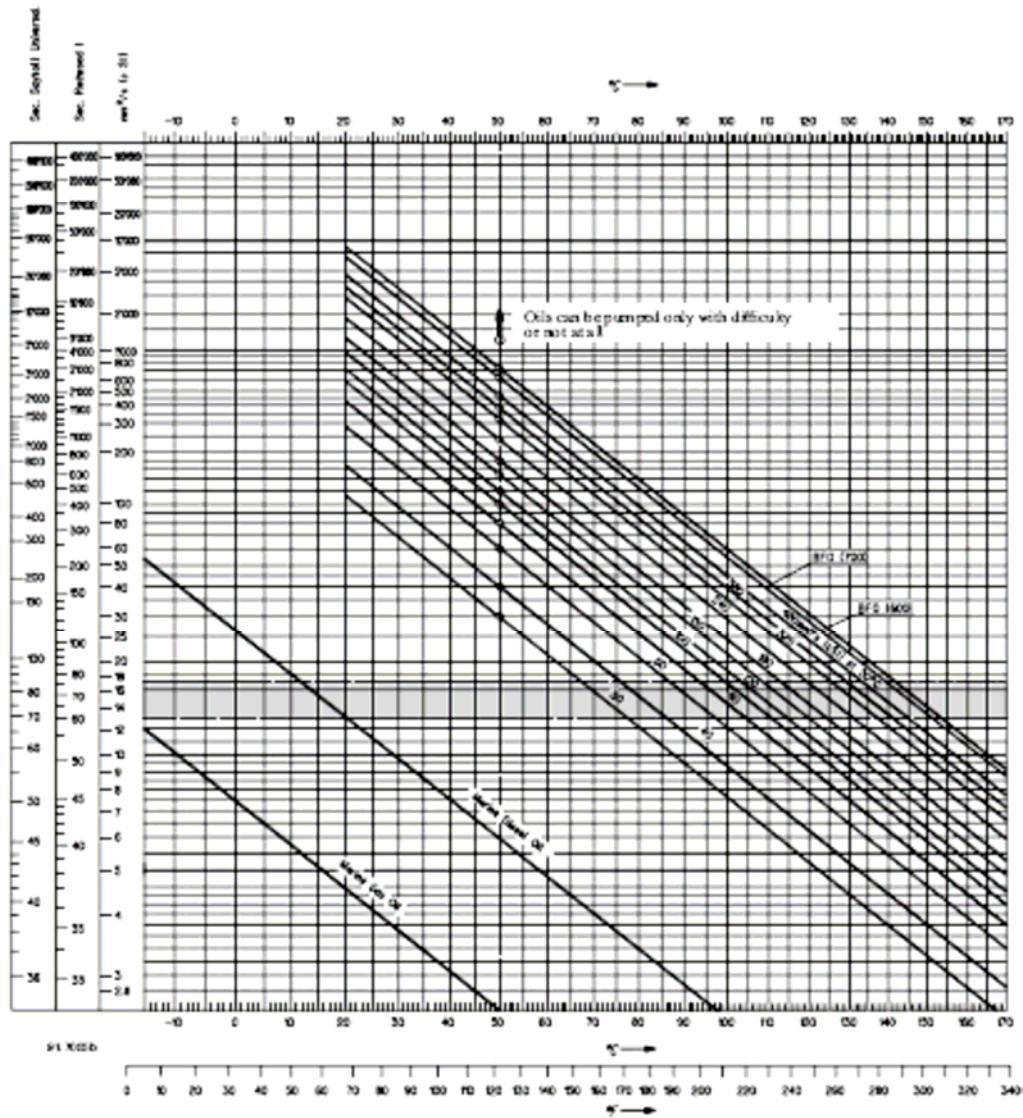
Other cavitation modeling efforts suggest that fuel viscosity is one of the most significant contributors to the potential incidence of cavitation (18). That is because the fuel viscosity is a dissipating property, thus higher viscosity fuels have a stronger dissipating effect on the

fluctuating pressure waves. It should be noted the viscosity of JP-5 is nominally 1.5 cSt at 40°C, versus nominally 3.0 cSt at 40°C for DFM. The recommended viscosity range for the 12ZAV40S engine is shown in Figure 45, and is recommended to be 13–17 cSt at the fuel injection pump inlet temperature. It is probably safe to assume the design and durability prototype of the fuel injection pump was performed with fuels in the aforementioned viscosity range. Thus fuels in the 1.5 to 3.0 cSt range could vary the cavitation sensitivity of the fuel injection system due to reduce dissipation. At this time it has yet to be determined if the increased operating speed utilized later in the test program may have contributed to the likelihood of the occurrence of cavitation, although the increased speed would have altered the turbulence of the fuel flow in the fuel injection pump body.

The image in Figure 46 from a commercial maritime fuel injection service company shows evidence of cavitation in a large engine barrel and plunger assembly, with the cavitation erosion being evident around one of the ports in the barrel. This image can be compared to Figure 47 that was taken after the barrel from the JP-5 fuels testing was sectioned. The erosion pattern around the port appears quite similar to the case documented in Figure 46, suggesting some cavitation occurred during testing. What has not been documented is whether the debris eroded due to cavitation can cause the scuffing seen during the JP-5 test program.

VISCOSITY - TEMPERATURE DIAGRAM

(Typical Values of Petroleum Fuel Oils)



Recommended viscosity before fuel injection pumps

Example:

To obtain the recommended viscosity before fuel injection pumps a fuel oil of 150 mm²/s (cSt) at 50°C must be heated to 108 to 120°C.

Service Bulletin Enclosure ZAS-22/ 5

Figure 45. Fuel Viscosity Recommendations

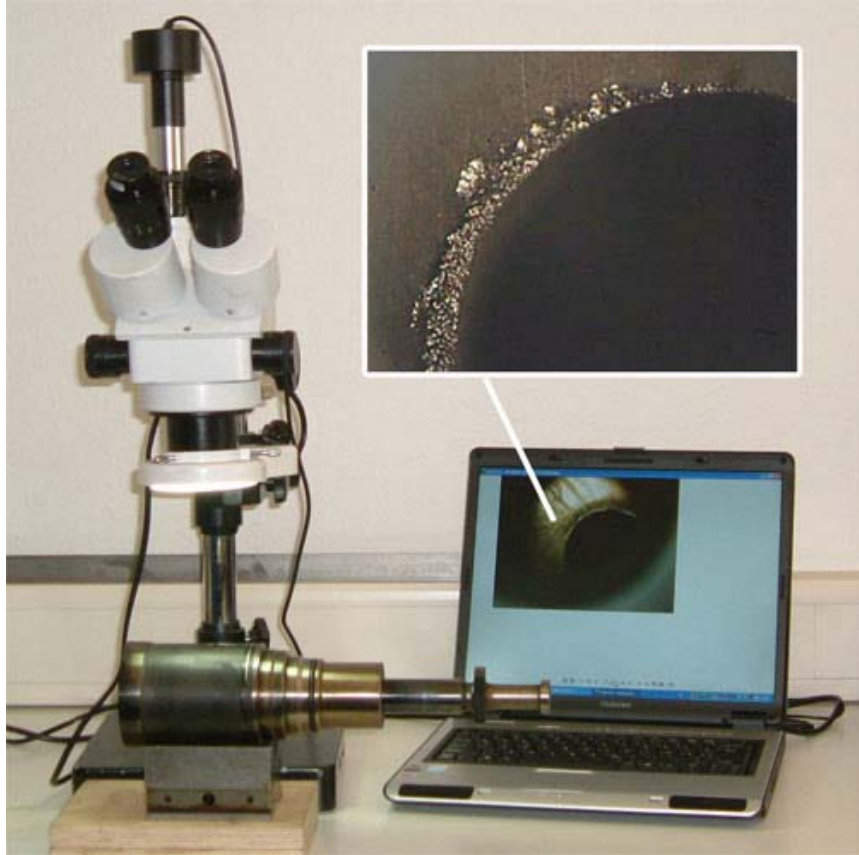


Figure 46. Documented Cavitation from Commercial Diesel Service Company

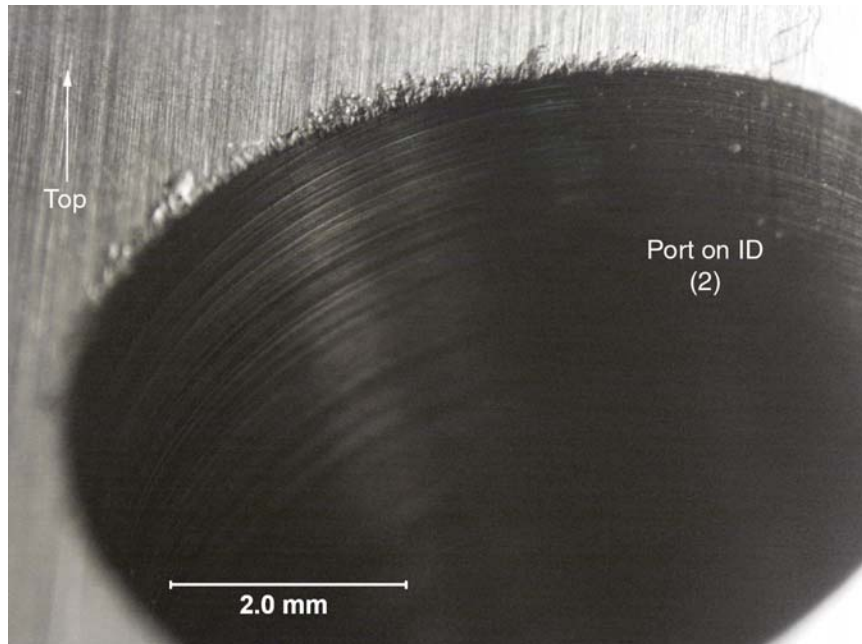


Figure 47. Barrel Port from JP-5 Test Program with Evidence of Cavitation-like Erosion

Due to complications, the radioactive tracer approach was not suitable to determine wear in the 12ZAV40S fuel injection system. Some type of controlled durability test would be required to determine suitability of JP-5. Indicators suggest short term JP-5 usage would likely be suitable, however evidence of cavitation erosion suggest long term use may be questionable. However the cavitation erosion seen was not documented for both fuels examined, nor is it certain whether the expanded speed operating conditions may have contributed to accelerating the cavitation erosion. Two approaches are suggested for determining wear with JP-5 without risk to the Healy and without experimental method altering the result:

- A controlled test of an operating engine where one cylinder is isolated to operate on JP-5 for a significant number of hours. Component condition would be compared to one of the other cylinder conditions. The components would be new at the start of testing and at the end of testing evaluated for condition. The components evaluated should be a barrel and plunger and a fuel injector. Fuel injectors are apparently more highly susceptible to cavitation erosion than the fuel injection pumps. A similar approach is detailed in SAE paper 2002-01-1700 that looked at a Wartsila VASA 9R32 diesel engine with varying lubricity fuels (19). The difficulty in this approach would be identifying a suitable engine for use or whether an engine on the Healy can be used to screen the fuels.
- A single pass pump rig test of a significant quantity of an aviation turbine type fuel, such that the effects of additive changes, fuels additions, component disassemblies and fuel/oil migration are not compounded. The wear on the barrel and plunger of the test fuel could be compared to an installed barrel and plunger and injector that has been removed from the Healy that was burning DFM. SwRI has an aviation fuel filtration facility for challenging large fuel distribution filters with 12,000-gallons of fuel in a single pass test mode. At 3L/min injection rate the rig could run for 250 hours without having to handle the fuel. The fuel could be clay treated, re-additized, and used again for a second pass to obtain 500-hour challenge to the fuel injection equipment. The fuel injection pump rig would be operated at the recommended engine speed, full-rack, and fuel inlet and outlet conditions and the lubricating oil inlet condition.

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