EVALUATION OF A PROPOSED F-4 EJECTION SEAT CUSHION BY +GZ IMPACT TESTS

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FOR THE COMMANDER

Thomas & Moore

THOMAS J. MOOŘE, Chief Biodynamics and Biocommunications Division Crew Systems Directorate Armstrong Laboratory

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PREFACE

The tests described within this report were accomplished by the Escape and Impact Protection Branch, Biodynamics and Biocommunications Division, Crew Systems Directorate of the Armstrong Laboratory. The vertical acceleration (impact) tests were conducted using a vertical deceleration tower to simulate the catapult phase of an aircraft ejection.

The impact facilities, data acquisition instrumentation, and data processing systems were operated by the Scientific Services Division of DynCorp under Air Force Contract F33615-86-C-0531.

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Special thanks to Mrs. Jeni Blake for her time and effort in organizing and preparing this report.

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INTRODUCTION

The objective of this test program was to measure the influence of specific ejection seat cushions on the human response to vertical acceleration. The test program, requested by the San Antonio Air Logistics Center, compared a proposed seat cushion and the survival kit lid designed to be used to an existing operational seat cushion and survival kit lid. Our study was designed to provide a direct comparison of the human response to vertical acceleration with these two seat cushions. The study also included, as a standard for comparison, a seat cushion that had been used in a previous comparative study (Hearon and Brinkley, 1986).

METHODS

Vertical impact tests were performed using a deceleration tower facility. The test conditions are shown in Table 1. The order of presentation of the test conditions was randomized for each subject. Other test conditions such as the seat geometry, restraint configuration, acceleration time history, and preimpact position of the subject, were controlled to assure that the measured responses were due only to the differences among the seat cushions and the seat surface on which they were tested.

TABLE 1. EXPERIMENTAL CONDITIONS										
MATRIX CELL DESIGNATION	ACCELERATION (G)	SEAT CUSHION	SEAT SURFACE							
F	10	ACES II	FLAT							
I	10	CURRENT F-4	CONTOURED							
J	10	PROPOSED F-4	FLAT							

Three seat cushions were evaluated. Two of the seat cushions were designed to be used with the F-4 ejection seat. One of the F-4 cushions was a current operational seat cushion manufactured by the Martin-Baker company (part no. MBEU 2866DP). The cushion, shown in Figure 1, consists of two layers of foam. The upper layer (part no. MBEU 2869DP) is a one-inch thick, heavy-density Texfoam[™] rubber. The lower layer (part no. MBEU 2870DP) is high-density, Plasazote[™] mold foam one-inch thick on each side. The lower layer is contoured on the top and bottom to fit the seat occupant's buttocks and the buttocks contour molded into the lid of a survival kit. In the area of the occupant's ischial tuberosities, the lower layer is 0.5-inch thick. The cushion is used in the U.S. Air Force with a fiberglass survival kit manufactured by Koch and Sons (part no. 140-489-1). The depth of the depression molded in the lid of the kit is 1.50 in. The cushion is covered with a stretchable Nomex[™] material (Fabratex Corp. part no. 1-771001).

The second cushion (Koch and Sons part no. 159-870286-1), which is also designed for the F-4 ejection seat, is constructed of uncontoured, two-inch thick, type C-47 ConformTM foam (Koch and Sons Part no. 159-870288-1) with a NomexTM cover (Koch and Sons part no. 144-870287-1). This cushion, shown in Figure 2, is designed to be used with a flat survival kit lid manufactured by H. Koch and Son (part no. 024-870280-1). Both the contoured lid and the flat lid are designed to be used with the same survival kit body (Koch and Sons part no. 140-710-1)/



FIGURE 1. OPERATIONAL F-4 EJECTION SEAT CUSHION



FIGURE 2. PROPOSED F-4 EJECTION SEAT CUSHION

The third cushion that was tested was an ACES II ejection seat cushion (see Figure 3). This cushion consists of three layers of material. The lower layer is 0.39 in (1 cm) thick, sheet foam polyethylene (DMS 1954, Class I, Grade 4101). The middle layer is 0.5 in (1.3 cm) thick type T-47 TemperTM foam. The upper layer consists of 0.25 in (0.6 cm) thick, space fabric (Uniroyal, 6007-1-1-54) to promote air circulation within the cushion. The ACES II cushion is covered with a stretchable NomexTM fabric.

The two F-4 ejection seat cushions were tested on their respective survival kit lids. The contents of the survival kits were assumed to have identical mechanical compression properties, and only differed in the contours of their lids. Each survival kit lid was bonded to a wooden block of identical composition. The kit lid and wooden block were then mounted to a metal plate, which was attached to the structure of the test seat by a force measurement device consisting of six force cells.

The ACES II seat cushion was tested on a flat seat supported by the force cells. The seat was constructed of a wooden block mounted to the metal plate used to attach the six force cells to the seat frame. The surface of the block was covered by a rigid plastic laminate material.

The test seat was the generic design shown in Figure 4. The seat back was positioned at right angles to the uninclined seat pan and parallel to the acceleration vector. The headrest contact plane was one inch (2.5 cm) aft of the seat back. The vertical position of the headrest was individually adjusted for each subject in order to provide adequate head support. This vertical headrest position was held constant for each subject during the test program. Seat-back cushions were not used in the study. Also, no footrest structure or leg restraint were provided, so the lower legs were permitted to dangle during the The subjects were restrained by the PCU-15/P torso/ test. parachute harness and an HBU-type lap belt constructed of 1.72in (4.37 cm) wide, type III polyester webbing (MIL-W-25361C). A MA-1 lap belt buckle was used. Before each test, the restraint system was pretensioned so that the force at the shoulder-strap and lap-belt attachments measured 20 \pm 5 lb (89 \pm 22 N). All subject wore the HGU-26/P flight helmet and were initially positioned with head upright, helmet against the headrest, and arms resting on anterior thighs.

The experimental set up and pre-test position of the subject are shown in Figure 5. The test fixture, restraint system, and subject were instrumented to obtain pertinent data during each test. Measured parameters included the translational acceleration of the deceleration tower carriage



FIGURE 3. ACES II EJECTION SEAT CUSHION







FIGURE 5. PRE-TEST POSITION OF THE TEST SUBJECT

and test seat, angular acceleration of the seat about the Y axis, velocity of the carriage, vertical and horizontal seat forces, and triaxial forces measured at the restraint harness attachment points. Triaxial translational accelerations and angular accelerations about the Y axis were measured at the head and chest of the subject. Photogrammetric data were obtained by two high-speed motion picture cameras mounted on the test fixture, permitting measurement of body displacements. The electronic and photogrammetric data acquisition system is described in detail in Appendix A.

The left-handed coordinate reference system for acceleration (+x anterior, +z cephadalad) was used during data analysis. Electronic and photogrammetric data were processed by a Digital Equipment Corporation PDP-11/34 computer.

The test results were evaluated using the Wilcoxon pairedreplicate rank test (Wilcoxon and Wilcox, 1964). This statistical technique was selected to compare the peak values of measured parameters and to establish the statistical significance of observed trends in the data. This analytical approach established each subject as his own control, thereby reducing the effects of biological variability among subjects. The 95th percent confidence level, assuming a two-tailed test, was chosen as the level of statistical significance.

The hypothesis that was statistically evaluated was that there are no differences between the measured human responses regardless of the cushion that is used. Evaluation criteria were based on the fundamental principles of biomechanical Clinically consequential impact injury generally protection. results from differential acceleration of body segments and/or excessive structural loading. For short-duration acceleration applied parallel to the spinal column, human tolerance is apparently limited by vertebral compression fracture. If the risk of such injury is to be reduced, vertebral column loading must be minimized during vertical acceleration. Minimizing head and chest acceleration would also be wise since they may be indicative of potentially injurious internal forces. Accordingly, the key response parameters in this study were the resultant seat force, which is generally reflective of vertebral column loading, and the resultant head and chest accelerations.

The volunteer subjects, 13 men, were active-duty officers and enlisted personnel at Wright-Patterson Air Force Base who were medically qualified for impact acceleration stress experiments. The subjects were required to meet stature, weight, and sitting height criteria for USAF pilots and a complete medical screening more stringent than the USAF flying class II evaluation. Conservative selection criteria were established to yield a subject sample comparable to the USAF flying population in terms of age and anthropometry, but super normal in terms of susceptibility to impact injury. The characteristics of the subject sample used in this study (in terms of mean and standard deviation) may be summarized as follows: age, 28.8 ± 4.13 years; weight, 174 ± 21.2 lb; height, 70.2 ± 2.70 in; and sitting height, 33.5 ± 1.31 in. These values compare favorably with the means and standard deviations for the USAF flying population (NASA Ref. Pub. 1024). The sizes and weights of the subjects are described and compared to the USAF flying population in Table 2.

TABLE 2. ANTHROPOMETRY OF TEST SUBJECTS										
SUBJ NO.	WT (LB)	STATURE (IN)	SITTING HEIGHT (IN)	MID-SHOULDER SIT HEIGHT (IN)	BUTTOCK TO KNEE (IN)	AGE (YR)				
B-1	160	70.5	37.1	25.7	23.9	28				
D-5	175	72.0	36.1	25.0	25.0	25				
L-3	190	72.0	37.4	26.1	25.1	36				
L-5	186	69.8	35.6	25.4	25.1	27				
M-16	199	70.0	37.3	25.7	24.6	32				
M-19	181	74.2	38.5	26.1	25.6	26				
M-20	199	70.8	37.0	26.0	25.0	29				
M-21	126	66.0	34.1	23.5	23.0	28				
0-2	178	65.7	34.0	24.0	23.6	26				
P-5	186	68.5	36.0	24.6	23.2	25				
R-8	169	74.6	37.6	26.3	26.4	28				
S-3	167	69.5	36.5	25.5	23.7	38				
Z-2	143	68.3	36.9	25.4	22.9	26				
TEST	SUBJE	CT MEANS	AND STANDAR	DEVIATIONS						
MEAN	173.8	70.2	33.5	25.3	24.4	28.8				
s.D.	21.2	2.7	1.31	0.85	1.09	4.13				
USAF	FLYIN	G PERSON	NEL (1967 SU	RVEY)		•				
MEAN	173.6	69.8	36.7	25.4	23.8	30.0				
s.D.	21.4	2.4	1.3	1.10	1.10	6.31				

The tests were conducted using presumed subinjury, shortduration acceleration conditions to minimize the potential for injury to the subjects. Following a low-level test to familiarize the participants with the test procedures and equipment, a randomized series of tests at a nominal 10 G peak and velocity change of 27 ft/sec (8 m/sec) were initiated. The acceleration profile was an approximate half-sine waveform, and the average time to peak carriage acceleration (rise time) was 70 msec.

The tests were carried out at the Armstrong Laboratory using the Vertical Deceleration Tower (VDT). The test assembly, including seat, restraint, and instrumentation, was mounted to the impact carriage of the VDT. This carriage was raised to a drop height of 10 ft 4 in (3.14 m) and then allowed to fall freely along vertical rails onto a hydraulic decelerator at the base of the tower. The vertical acceleration was produced when a plunger fixed to the bottom of the carriage displaced water in the hydraulic decelerator. To assure identical acceleration conditions, the carriage drop height, test assembly mass, water volume, and plunger type were the same for all experimental-level tests in this study.

RESULTS

The test results are summarized in Table 3 in terms of the means and standard deviations of the measurements for each of the three cushions that were evaluated. Appendix B provides typical sets of electronic data from tests of each cushion and he maxima and minima of each measurement from each test.

TABLE	3. SUMMARY OF	DATA	
RESPONSE PARAMETER	CELL F ACES II	CELL I CURRENT F-4	CELL J PROPOSED F-4
	n = 13	n = 13	n = 13
SEAT ACCELERATION (G)	10.64 <u>+</u> 0.38	10.82 <u>+</u> 0.22	10.80 ± 0.29
RESULTANT SEAT FORCE (LB)	2698 <u>+</u> 362	2775 <u>+</u> 349	2639 <u>+</u> 342
SEAT FORCE RISE TIME (SEC)	0.079 <u>+</u> 0.007	0.083 <u>+</u> 0.004	0.082 ± 0.005
RESULTANT CHEST ACCELERATION (G)	16.32 <u>+</u> 2.15	17.53 <u>+</u> 1.98	16.17 <u>+</u> 1.36
CHEST ACCELERATION RISE TIME (SEC)	0.079 <u>+</u> 0.005	0.083 <u>+</u> 0.006	0.078 <u>+</u> 0.003
RESULTANT HEAD ACCELERATION (G)	13.76 <u>+</u> 1.29	15.38 <u>+</u> 1.17	13.42 <u>+</u> 0.95
HEAD ACCELERATION RISE TIME (SEC)	0.077 <u>+</u> 0.005	0.080 <u>+</u> 0.004	0.078 <u>+</u> 0.003
RESULTANT SHOULDER-STRAP FORCE (LB)	191 <u>+</u> 72.5	178 <u>+</u> 58.3	188 <u>+</u> 52.7
LEFT LAP-BELT FORCE (LB)	110 <u>+</u> 35.0	99 <u>+</u> 33.6	103 + 19.7
RIGHT LAP-BELT FORCE (LB)	104 <u>+</u> 25.0	91 <u>+</u> 31.4	97 <u>+</u> 23.7

The results of each set of Wilcoxon comparisons are summarized in Table 4.

TABLE 4. WILCOXON COMPARISON SUMMARY									
RESPONSE PARAMETER	CURRENT F-4 (I) PROPOSED F-4 (J)	ACES II (F) PROPOSED F-4 (J)	CURRENT F-4 (I) ACES II (F)						
	n = 13	n = 13	n = 13						
RESULTANT SEAT FORCE (LB)	I > J 99%	F = J	I > F 99%						
RESULTANT CHEST Acceleration (G)	I>J 98%	$\mathbf{F} = \mathbf{J}$	I > F 99%						
RESULTANT HEAD Acceleration (G)	I > J 99%	$\mathbf{F} = \mathbf{J}$	I > F 99%						

Percentages designate statistical confidence level

For a given comparison, a greater-than or less-than symbol designates a statistically significant difference in the response parameter at the chosen 95 percent confidence level; the symbol also indicates the direction of the trend, i.e., whether it increases or decreases in value from one cell to the other.

The seat acceleration for all 39 experimental-level tests in this study was well controlled during the evaluation. The maximum seat accelerations for each replicate set of tests were statistically evaluated for differences using the Wilcoxon analysis. Statistically significant differences were not found.

Direct comparisons of the two F-4 cushions are shown in the first comparison listed in Table 4. The evaluation parameters of seat force, head acceleration, and chest acceleration were all lower when the proposed F-4 cushion was used. The comparison of the proposed F-4 cushion with the ACES II cushion, shown in the second column of matched pairs, revealed no statistically significant differences. Comparison of the operational F-4 cushion with the ACES II, shown in the third column of matched pairs, revealed statistically significant differences in all of the evaluation parameters with lower values being measured with the ACES II cushion.

Each of the test conditions were well tolerated by the volunteer subjects. There were no injuries resulting from any of the tests.

DISCUSSION

The current operational seat cushion was selected for USAF use as a result of a series of ejection tests by Brinkley et al (August 1967) using a Martin Baker ejection seat with reducedcharge catapults. Three cushion conditions were evaluated: the Martin Baker cushion, a cushion developed by the McDonnell-Douglas Corporation, and no seat cushion. Forty-nine tests were conducted with seven human subjects at acceleration levels ranging from 6 to 14 G. The principal measurement that was used to evaluate the relative merits of the three cushion conditions was the vertical force between the accelerating seat and a simulated survival kit lid upon which a volunteer subject was seated. Typical data acquired from the ejection tests are plotted in Figure 6. Figure 6 is a plot of the ejection force (the measured seat acceleration times the subject mass) versus the vertical force measured between the seat and the survival kit lid. The seat acceleration level for this test was 9.9 G.

Although these tests more closely simulated actual F-4 ejection seat accelerations and seat geometries, the tests were more costly in time, funding, and injuries. Three of the seven subjects who participated in the tests incurred cervical, thoracic, and lumbar-sacral paravertebral sprains. One subject incurred a fracture of the spinal coccyx.

Analysis of the data collected during the earlier series of tests showed that seat force, the measured response used to comparatively evaluate the cushions, reached its critical value within about 0.10 sec to 0.12 sec.

This relationship can be clearly seen in Figure 6. Thus, one can conclude with confidence that only the first 0.1 sec of the catapult acceleration was crucial to the cushion evaluation. Furthermore, although the earlier tests were accomplished at acceleration levels from 6 to 14 G, the beneficial effects of the Martin Baker seat cushion could be seen at a level of 8.7 G. These results have led us to conclude that simpler tests at acceleration levels that represent a lower risk of injury to the volunteers will provide equivalent results.

Although the influence of the acceleration-transmission characteristics of an ejection seat is crucial, other factors should also be seriously considered. These factors include durability, flammability toxicity of gases produced during combustion, long-duration crew comfort, changes of mechanical properties as a function of temperature and age, and changes in the seat occupant's position that might influence the effectiveness of the cockpit layout or escape system



FIGURE 6. PLOT OF MEASURED FORCE AND COMPUTED FORCE FROM F-4 EJECTION SEAT TEST WITH HUMAN SUBJECT

performance. If any or all of these factors represent critical issues to the operational application, they should be thoroughly evaluated.

The long-duration comfort of the ConforTM foam has been demonstrated by tests in the F/FB-111 aircraft, and more recently by long-duration flights of the B-1B bomber. The thickness of the F/FB-111 cushions and B-1B cushions that were tested is identical to the proposed F-4 cushion that was tested by our Laboratory. Both the F/FB-111 and B-1B crew seats are contoured, although the contours are different. The B-1B seat cushion has been tested with the production contoured seat and with a flat seat. No significant difference was noticed by the crew; therefore, a decision was made by the B-1 System Program Office to use the cushion with the existing contoured seat.

CONCLUSION

The results of these tests show that the human response to vertical acceleration was more benign when either the proposed F-4 seat cushion or the ACES II seat cushion was used instead of the current operational F-4 seat cushion. From an acceleration protection standpoint, the operational F-4 cushion is inferior to the proposed alternative cushion or the operational ACES II cushion.

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TEST CONFIGURATION AND DATA ACQUISITION SYSTEM FOR THE EFFECTS OF SEAT CUSHIONS AND SEAT BACK ANGLE ON HUMAN RESPONSE DURING +Gz IMPACT ACCELERATION

TEST PROGRAM

Prepared under Contract F33615-86-C-0531 November 1986

Prepared by DynCorp (formerly Dynalectron Corporation) AAMRL Division Building 824, Area B Wright-Patterson AFB, Ohio 45433

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INTRODUCTION

This report was prepared by DynCorp (formerly Dynalectron Corporation) for the Harry G. Armstrong Aerospace Medical Research Laboratory (AAMRL/BBP) under Air Force Contract F33615-86-C-0531.

The information provided herein describes the test facility, test fixture, data acquisition, instrumentation procedures and and the test configuration that were used in "The Effects of Seat Cushions and Seat Back Angle on Human Response During +Gz Impact Acceleration Test Program." The testing was done on the Vertical Deceleration Tower starting July 1986 and ending August 1986.

1. TEST FACILITY

The AAMRL Vertical Deceleration Tower, as shown in Figure A-1, was used for all of the tests.

The facility consists of a 60 ft. vertical steel tower which supports a guide rail system, an impact carriage supporting a plunger, a hydraulic deceleration device and a test control and safety system. The impact carriage can be raised to a maximum height of 42 ft. prior to release. After release, the carriage free falls until the plunger, attached to the undercarriage, enters a water filled cylinder mounted at the base of the tower. The deceleration profile produced as the plunger displaces the water in the cylinder is determined by the free fall distance, the carriage and test specimen mass, the shape of the plunger and the size of the cylinder orifice. For these tests, plunger number 102 was mounted under the carriage. Drop height varied depending on the test cell requirements which ranged from 5'6" to 8'3".

2. SEAT FIXTURE

The VIP seat fixture, as shown in Figure A-1, was used for all of the tests. The seat was designed to withstand vertical impact acceleration up to 50 Gs. Its adjustable seat back allowed the subject to sit in one of four positions, as shown in Figure A-2. When positioned in the seat, the subject's upper legs were bent 90 degrees outward to a horizontal position with his lower legs bent 90 degrees downward to a vertical position. The subject was secured in the seat with a lap belt and shoulder strap. The lap belt and shoulder strap were preloaded to 20+lbs. as required in the test plan.

3. INSTRUMENTATION

The electronic data collected during this test program is described in Sections 3.1 and 3.2. Section 3.1 discusses accelerometers while Section 3.2 discusses load transducers. Section 3.3 discusses the calibration

procedures that were used. The measurement instrumentation used in this test program is listed in Tables A-la through A-lc. These tables designate the manufacturer, type, serial number, sensitivity and other pertinent data on each transducer used.

Accelerometers and load transducers were chosen to provide the optimum resolution over the expected test load range. Full scale data ranges were chosen to provide the expected full scale range plus 50% to assure the capture of peak signals. All transducer bridges were balanced for zero output except for those accelerometers in line with the force of gravity which were adjusted for plus 1 G. The accelerometer and load transducer coordinate system is shown in Figure A-3.

The accelerometers were wired to provide a positive output voltage when accelerations were applied in the +x, +y and +z directions, as shown in Figure A-3.



1. Typical fixed load cell and load link mounting point. Direction of arrows indicate direction of force applied to produce a positive output.

NOTE: Accelerometers were wired to produce a positive output voltage when accelerations were applied in the +x, +y and +z directions as shown.

FIGURE A-3: AAMRL/BBP COORDINATE SYSTEM

A-5

The load transducers included three types of load measurement devices. All were wired as follows:

Fixed Load Cells - were wired to provide a positive output when force is applied in the indicated direction (Figure A-3).

Triaxial Load Cells - were wired to provide a positive output when the belt was pulled towards the center of the seat.

Load Links - were wired to provide a positive output when force is applied in the direction indicated (Figure A-3).

Carriage velocity was measured using a Globe Industries tachometer Model 22A672-2. The rotor of the tachometer was attached to an aluminum wheel with a rubber "O" ring around its circumference to assure good rail contact. The wheel contacted the track rail and rotated as the carriage moved, producing an output voltage proportional to the velocity.

3.1 Accelerometers This section describes the accelerometer instrumentation as required in the AAMRL/BBP test plan.

Head accelerations were measured using three Endevco Model 2264-200 linear accelerometers and one Endevco Model 7302A angular (Ry) accelerometer. The accelerometers were mounted to the external edge of a plastic dental bite block. Each subject had his own set of custom fitted dental inserts that were used to support the bite block in his mouth. An aluminum tube extended from the bite block and located a fiducial target used for photo tracking purposes.

The chest accelerometer package consisted of three Endevco Model 2264-150 linear accelerometers mounted to a $1/2 \times 1/2 \times 1/2$ inch aluminum block. An Endevco Model 7302A angular (Ry) accelerometer was mounted on a bracket adjacent to the triaxial chest block. The accelerometer packages were inserted into a steel protection shield to which a length of Velcro fastener strap was attached. The package was placed over the subject's sternum at the level of the xiphoid and was held there by fastening the Velcro strap around the subject's chest.

Carriage accelerations were measured using three Endevco accelerometers: Model 2262A-200 for the z direction, Model 2264-200 for the x direction and Model 7264-200 for the y direction. The three accelerometers were mounted on a small acrylic block and located behind the seat on the VIP seat structure.

Seat accelerations were measured using three Endevco accelerometers: one Model 2264-150 for accelerations in the x direction and two Model 2264-200s for accelerations in the y and z directions. Seat angular (Ry) acceleration was measured using an Endevco Model 7302B angular accelerometer. The three linear accelerometers were attached to a 1 x 1 x 3/4 inch acrylic block and were mounted below the seat near the back edge of the support frame. The angular accelerometer was attached to an aluminum bracket and was mounted near the center and below the seat.

Head accelerations for dummy tests were measured using three Endevco Model 2264-200 linear accelerometers and one Endevco Model 7302 angular accelerometer. These accelerometers were internally mounted in the head of the VIP 95 manikin.

3.2 Load transducers This section describes the load transducer instrumentation as required in the AAMRL/BBP test plan.

The load transducer locations and dimensions are shown in Figure A-4.

Right lap, left lap and shoulder strap loads were each measured using GM3D-SW triaxial load cells, each capable of measuring loads in the x, y and z directions. The shoulder strap triaxial package was mounted on the seat frame between the seat back support plate and the headrest. The right and left lap triaxial packages were located on separate plates mounted on the side of the seat frame parallel to the seat pan.

Seat pan loads were measured using three load cells and three load links. The three load cells were Strainsert Model FL2.5U-2SPKT load cells. The three load links, as shown in Figure A-5, were fabricated by DynCorp using Micro Measurement Model EA-06-062TJ-350 strain gages. All six measurement devices were located under the seat pan support plate. The load links were used for measuring loads in the x and y directions, two in the x direction and one in the y direction. Each load link housed a swivel ball which acted as a coupler between the seat pan and load cell mounting plate. The Strainsert load cells were used for measuring loads in the z direction.



FIGURE A-5: LOAD LINK INSTRUMENTATION

A-7

3.3 Calibration

Calibrations were performed before and after testing to confirm the accuracy and functional characteristics of the transducers. Pre-program and post-program calibrations are given in Tables A-2a through A-2d.

The calibration of all Strainsert load cells was performed by the Precision Measurement Equipment Laboratories (PMEL) at Wright-Patterson Air Force Base. PMEL calibrated these devices on a periodic basis and provided current sensitivity and linearity data.

The calibration of the accelerometers was performed by DynCorp using the comparison method (Ensor, 1970). A laboratory standard accelerometer, calibrated on a yearly basis by Endevco with standards traceable to the National Bureau of Standards, and a test accelerometer were mounted on a shaker table. The frequency response and phase shift of the test accelerometer was determined by driving the shaker table with a random noise generator and analyzing the outputs of the accelerometers with a PDP 11/15 computer and 1923 Time Data Unit using Fourier analysis. The natural frequency and the damping factor of the test accelerometer were determined, recorded and compared to previous calibration data for that test accelerometer. Calibrations were made at 40 G and 100 Hertz. The sensitivity of the test accelerometer was determined by comparing its output to the output of the standard accelerometer.

The angular accelerometers were calibrated by DynCorp by comparing their output to the output of a linear standard accelerometer. The angular accelerometer is mounted parallel to the axis of rotation of a Honeywell low inertia D.C. motor. The standard accelerometer is mounted perpendicular to the axis of rotation at a radius of one inch to measure the tangential acceleration. The D.C. motor motion is driven at a constant sinusoidal angular acceleration of 100 Hertz and the sensitivity is calculated by comparing the rms output voltages of the angular and linear accelerometers.

The velocity wheel was calibrated by rotating the wheel at various revolutions per minute (RPM) and recording both the output voltage and the RPM. The sensitivity was dynamically checked with a G-HI measuring system and the Horizontal Accelerator Sled facility. This system consists of a timing unit and an optical sensor mounted near the track rails. As the sled traveled along the track rails, a metal blade on the sled interrupted the optical sensor beam. The timing unit displayed a time which was correlated to a velocity.

The load links and GM load cells were calibrated by DynCorp. These transducers were calibrated to a laboratory standard load cell in a special test fixture. The sensitivity and linearity of each test load cell were obtained by comparing the output of the test load cell to the output of the laboratory standard under identical loading conditions. The laboratory standard load cell, in turn, is calibrated by PMEL on a periodic basis.

4. DATA ACQUISITION

Data acquisition was controlled by a comparator on the Master Instrumentation Control Unit in the Instrumentation Station. The comparator was set to start data collection at a preselected time. A reference mark was electronically initiated to mark the electronic data and initiate a stobe light in the test area to mark the film frame for reference. The test was initiated when the countdown clock reached zero. The reference mark, used in the processing of data, was generated after $T = \emptyset$ to place the reference mark close to the impact point.

Timing reference was provided by a master clock. Timing pulses of 100 pps were provided by the master clock to film data. The cameras were run at 500 frames per second and a timing pulse was placed on the film at 10 millisecond intervals.

Prior to each test and prior to placing the subject in the seat, data was acquired to establish a zero reference for all data sensors. This data was stored separately from the test data and was used in the processing of data.

4.1 Automatic Data Acquisition and Control System (ADACS) Installation of the ADACS instrumentation is shown in Figure A-6. The three major components of the ADACS system are the power conditioner, signal conditioners and the encoder. A block diagram of the ADACS is shown in Figure A-7. The signal conditioners contain forty-eight module amplifiers with programmable amplifier gains and filters.

Bridge excitation for load cells and accelerometers was 10 VDC. Bridge completion and balance resistors were added as required to each module input connector.

The forty-eight module output data signals were digitized and encoded into forty-eight 11-bit digital words. Two additional 11-bit synchronization (sync) words were added to the data frame making a fifty word capability.

Three synchronization pulse trains (bit sync, word sync and frame sync) were added to the word frame and sent to the computer via a junction box data cable.

The PDP 11/34 mini-computer received serial data from the ADACS. The serial data coming from the carriage were converted to parallel data in the data formatter. The data formatter inputs data by direct memory access (DMA) into the computer memory via a buffered data channel where data were temporarily stored on disk and later transferred to magnetic tape for permanent storage. The interrelationships among the data acquisition and storage equipment are shown in Figure A-8.

Test data could be reviewed immediately after each test by using the "quick look" SCAN routine. SCAN was used to produce a plot of the data

stored on any channel as a function of time. The routine determined the minimum and maximum values of any data plot. It was also used to calculate the rise time, pulse duration and carriage acceleration.

4.2 Photogrammetric Data Acquisition

Two onboard high-speed LOCAM cameras, operating at 500 frames per second, were used to produce the photogrammetric data. Each camera used a 9mm lens and were automatically started at a preset time in the test sequence by a signal from the camera and lighting control station. Both camera locations are shown in Figure A-9.

Motion of the subjects' head, shoulders and chest were quantified by tracking the motion of subject-mounted fiducials. Reference fiducials were placed on the test fixture. Two different sized fiducials were used, one being a .75" diameter black circle on a 1.25" diameter white target, the other a 1.25" diameter black circle on a 2.00" diameter white target. The locations of the fiducials generally followed the guidelines provided in "Film Analysis Guides for Dynamic Studies of Test Subjects, Recommended Practice" (SAE J138, March 1980). Fiducial target locations are identified in Figure A-10.

The photogrammetric data were time correlated in each test. Immediately prior to impact, an event signal triggered the flash unit to mark the camera film frame. At that time, a 100 PPS signal activated the camera L.E.D. driver which pulsed the camera L.E.D., producing a time mark at the film edge. This reference mark was then used to correlate the photogrammetric data with the electronically measured data.

The photogrammetric data will be processed as required on the Automatic Film Reader (AFR) system, shown in the block diagram in Figure A-11. The fiducial tracking routine is initiated via the Data General terminal. The tracking routine is booted from a floppy disk into the Nova 3/12 memory. The system is capable of tracking fiducials manually or automatically. The Nova 3/12 outputs an x-y film coordinate position to magnetic tape for each fiducial being tracked. Data are then transferred from magnetic tape to the DEC PCP 11/34 disk file for processing.

An Instant Analytical Replay (INSTAR) video system was also used to provide photogrammetric coverage of each test. This video recorder and display unit is capable of recording high-speed motion at a rate of 120 frames per second. Immediate replay of the impact is possible in real time or in slow motion.

5. PROGRAM OPERATION

5.1 Introduction

This section identifies the flowcharts and processing programs that were used for the VSBA Study conducted by the Biomechanical Protection Branch, Biodynamics and Bioengineering Division of the Harry G. Armstrong Aerospace Medical Research Laboratory.
The executable tasks for the VSBA Study processing programs are located on the Data Processing disk of the PDP 11/34. The test data is read into the computer using the DEC Peripheral Interchange Package from a digital magnetic tape with a density of 800 BPI and stored on an RL \emptyset 2 hard disk. All plots are output to a Tektronix hardcopy unit. The alphanumeric data itself is output to the Versatec line printer.

5.2 Program Operation

The five Fortran programs that process the VSBA Study test data are named "VSVDØA," "VSVDØB," "VSVDØC," "VSVDØD" and "VSVDØE." The command file which controls execution of these tasks is named "VSVD." The two characters "VS" identify the study (VSBA), the characters "VD" identify the facility (Vertical Deceleration Tower), "Ø" is the revision number and the last character determines the program order of execution.

Task A requires the user to enter the total number of tests to be processed and the zero and data filenames for each test. The user must then specify whether the default test parameters are to be used for processing. If the default parameters are selected, then the test number, subject identification, weight, age, height and sitting height are read in from the first block of the test data file. The cell type, nominal G level and left lap, right lap and shoulder preload values are also read in. If the default parameters are not selected, they must be entered by the user. Task A creates a command file containing execution commands for each test, which is called by command file "VSVD" after task A exits.

Task B creates the individual data files for each channel and data files for all sums, differences, products and resultants. Task C finds data maxima and minima for each channel, does any special processing required and outputs results to the data base. Task D outputs an alphanumeric cover sheet to the Versatec line printer/plotter based on the formats specified in the base and report format files. Task E plots the specified data channels for 600 ms after the reference mark and hardcopies the plots.

5.3 Program Flowcharts

Flowcharts of the five programs are shown in Figures A-12 through A-16. Each flowchart identifies the files used and the subroutines called by the program. Data channel numbers and accelerations or loads are listed where they occur in the analysis.

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-	Carriage Y	Endevco 7264-200	AT CHA	2.767 w/a	10.01	3	100		9.04 8	120	2.2	270K tin Cad	:		
•	Head X	Endevco 2264-200	8756	2.821 av/8	10.00	3	× _	×	17.7 .	120	2.2 5.0	•	1.65K		
•	Head T	*	C723	2.224 **/8	10.00	30	2 92	IK I	22.4 g	120	2.5	130K +in Cad	r		
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•	Cheet X	Endevco 2264-150	BC26	2.807 27.6	10.00	\$,1 ,1	1	17.6 g	120	2.5	1.2N - la Cad	2		
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•	Cheet 2	:	2A20	2.648 av/a	0.00	° °	2	- ¥	3 1.16	120	2.5 0.00 0.00	155K -in Ced	Ŧ		
9]	Left Seat Load	Straineer FL2.5U- ZSPET	1-4626	8.02 uv/1b	10.00	10	102	-	1351 LB	120	2.5	•	•		
=	Right Seat Load	1	3294-2	8.03 4v/16	10.00		201		81 8451	120	2.5 0.0 0.0	,	,		
51	Center Seat Load	:	3294-4	8.08 uv/16	10.00	12	8	- -	3094 La	971	2.0	•	1		
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TABLE A-1a: INSTRUMENTATION REQUIREMENTS (PAGE 1 OF 3)

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THE EFFE	VERTICAL	BATA POINT	Left Logd X	Left Lep Load i	Left Lep Load 2	Right Lap Load X	Right Lap Load Y	Right Lep Load 2	Showlder Load X	Shoulder Load Y	Shoulder Load 2	Seat X Accel.	Seat Y Accel.	Seet Z Accel.	Center Load Link Y	Vel. Tach.	145 - SEAT LOCA
ROORAM	ACHITY	BATA CHANNEL	2	16	17	8	2	20	21	2	2	25	26	27	26	58	12.87 1

TABLE A-1b: INSTRUMENTATION REQUIREMENTS (PAGE 2 OF 3)

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н			Phiten H E	2000	120	120	120	2000		360	360	120	120	120	120			
AUC B6			7.4. MM	5.0 Volt	2997 RAD/SEC ²	1862 RAD/SEC ²	1704 RAD/SEC ²	s.o Vote		2.5 Volt	2.5 Valt	17.2 6	18.2 8	37.6	3063 RAD/SEC ²			
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TABLE A-1c: INSTRUMENTATION REQUIREMENTS (PAGE 3 OF 3)

THE EFFECTS OF SEAT CUSHION AND PROGRAM: <u>SEAT BACK ANGLES DU</u>RING +Gz DATES: <u>14 JUL 86 - 14 AUG</u>.86

FACILITY: VERTICAL DECELERATION RUN NUMBERS: 1127-1214

	TOWER
-	

			-384	-CAL	POST	-CAL		
DATA POINT	MFG. & MODEL	NUMBER	DATE	* SENS	DATE	* SENS	K CHANGE	NOTES
CARRIAGE Z	ENDEVCO 2262A-200	FR42	7JUL86	4.20	20AUG86	4.189	3	* ALL SENS. IN mv/g
CARRIAGE X	ENDEVCO 2264-200	BX17	8JUL86	2.792	:	2.804	+.4	
CARRIAGE Y	ENDEVCO 7264-200	вн97н	9JUL86	2.767	=	2.783	+ .6	
HEAD X	ENDEVCO 2264-200	BP56	28MAY86	2.821	14AUG86	2.839	9.+	
HEAD Y	£	CF23	=	2.224	Ŧ	2.257	+.4	
HF.AD Z	÷	СН73	:	2.741	=	2.751	+1.5	
CHEST X	ENDEVCO 2264-150	BC26	Ŧ	2.807	:	2.795	4	
CHEST Y	Ŧ	BB13	r	2.467	E	2.438	-1.2	
CHEST Z	F	2A20	E	2.648	ĩ	2633	6	
SEAT X	=	BB 2:8	31DEC85	2.700	20AUC86	2.701	0	

PAGE 1 OF 4

TABLE A-2a: TRANSDUCER PRE- AND POST-CALIBRATION (1 OF 4)

SEFTERBUR 1985

DATES: <u>14 JUL 86 - 14 AUG</u> 86 RUN NUMBERS: 1127-1214 THE EFFECTS OF SEAT CUSHION AND PROGRAM: SEAT BACK ANGLES DURING +Gz FACILITY: VERTICAL DECELERATION

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		NOTES	* ALL SENS. IN mv/g UNLESS	NOTED OTHER- WISE.								
ſ		* CHANGE	+.1	£.+	+1.1	ο	+ .5	+.5	+1.0	+2.2	+1.8	
	-c.	* SENS	2.987	2.820	2.939	2.740	2.676	8.203 uv/RAD /SEC	4.195 uv/RAD /SEC	3.732 uv/RAD /SEC	6.80 uv/RAI /SEC	
	POST	DATE	20AUG86		22AUG86	I	F	=	15AUG86	22AUG86	15AUG86	
	-CAL	* SENS	2.985	2.812	2.906	2.740	2.662	8.163 uv/RAD /SEC	4.15 uv/RAD /SEC	3.65 uv/RAD /SEC2	6.68 uv/RAD /SEC2	
	- BRG	DATE	28MAY86	9JAN86	28MAY86	±	T	29MAY 86	28MAY86	6JAN86	29MAY86	
		NUMBER	BV95	BW07	CH74	BQ42	СН 70	A150	AB12	PT47	AB15	
		TRANSDUCEN MFG. & NODEL	ENDEVCO 2264-200	E	=	Ξ	Ξ	ENDEVCO 7302	ENDEVCO 7302A	ENDEVCO 7302B	ENDEVCO 7302A	
		DATA POINT	SEAT Y	SEAT Z	DUMMY IIEAD X	DUMMY IIEAD Y	DUMMY HEAD Z	DUMMY HEAD ANG.	HEAD ANG.	SEAT ANG.	CHEST ANG.	

TABLE A-2D: TRANSDUCER PRE- AND POST-CALIBRATION (2 OF 4)

PAGE 2 of 4

SEPTEMBER 1985

THE EFFECTS OF SEAT CUSHION AND PROGRAM:SEAT BACK ANGLES DURING +Gz DATES: 14 JUL 86 - 14 AUG 86

FACILITY & VERTICAL DECELERATION TOWER

TON RUN NUMBERS: 1127-1214

PAGE 3 OF 4

SEPTEMBER 1985

DATES: 14 JUL 86 - 14 AUG 86 RUN NUMBERS: 1127-1214 THE EFFECTS OF SEAT CUSHION AND **PROGRAM:** SEAT BACK ANGLES DURING +Gz DA FACILITY: VERTICAL DECELERATION DIN NUMB

3	2		KUN NURBERN	
		TOWER		

							_	_
	NOTES	* ALL SENS. IN uv/lb						
	& CHANGE	0	+.7					
CAL	* SENS	5.78	5.61					
1504	DATE	26AUG86	I					
-CAL	* SENS	5.78	5.57					
384	DATE	9JUL86	I					
	NUMBER	20Y	20 X					
Tourchicte	NFG. & NOPEL	GM 3D-SW	F					
	DATA POINT	SHOULDER LOAD Y	SHOULDER LOAD Z					

TABLE A-2d: TRANSDUCER PRE- AND POST-CALIBRATION (4 OF 4)

PAGE 4 OF 4

SEPTEMBER 1985



FIGURE A-1: VERTICAL DECELERATION TOWER

A-19

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MINUS 5° SEAT





	DESCRIPTION	DIMENSI	ONS IN CE	NTIMETERS	
		×	۲	2	
*1	SEAT REFERENCE POINT	0.00	0.00	0.00	
2	CENTER SEAT LOAD	+11.75	0.00	-7.94	
3	RIGHT SEAT LOAD	+40.64	+17.78	-7.94	
4	LEFT SEAT LOAD	+40.64	-17.78	-7.94	
5	LEFT LAP LOAD	-3.81	-22.86	-4.29	
6	RIGHT LAP LOAD	-3.81	+22.86	-4.29	
**7	SHOULDER STRAP LOAD	-14.67	0.00	+70.17	
8	CENTER SEAT LINK	+17.78	+5.08	-9.45	
9	RIGHT SEAT LINK	+20.32	+12.70	-9.45	
10	IFFT SFAT ITNE	+20 32	-12 70	-9 45	

* All dimensions are referenced to the seat reference point (SRP). The seat reference point is located at the intersection of the horizontal seat plate (x axis) center line and the vertical back plate (z axis) center line.

** Dimensions shown are for the seat in the zero degree position. x and z measurements varied for each different seat position.







FIGURE A-6: ADACS INSTALLATION







FIGURE A-8: DATA ACQUISITION AND STORAGE SYSTEM BLOCK DIAGRAM



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FIGURE A-9: ONBOARD CAMERA LOCATIONS



Fiducial target locations 6, 8 and 11 vary with respect
to seat back angles. Below are x, y and z dimensions
for seat back angles of +5, -5 and -10 degrees.

	DESCRIPTION	DIMENS	IONS IN FEE	т	D
		×	Ľ	<u>z</u>	6
1	UPPER HELMET	-	-	-	+5 ⁰ 8
2	FRONTAL HELMET	-	-	-	11
3	CHEEK POINT	-	-	-	
4	MOUTH PACK	-	-	-	6
5	CHEST PACK	-	-	-	-5 ⁰ 8
* 6	HEAD REST	-0.2146	+0.5531	+2.7028	11
7	UPPER PLATE	-1.1780	+0.6846	+2.1052	
* 8	LOWER PLATE	-0.6779	+0.6637	+1.8198	6
9	CAMERA STRUT	+1.7294	+2.3987	+2.2508	-10 ⁰ 8
10	CARRIAGE	-1.3238	+0.6429	+0.9167	11
*11	SIDE RAIL	-0.5273	+0.5635	+0.9807	
12	CENTER HELMET	-	-	-	A11 (SP

	DESCRIPTION	DIMENS	IONS IN FEE	T	
		x	X	<u>2</u>	
	6 HEAD REST	-0.4465	+0.5531	+2.6823	
5°	8 LOWER PLATE	-0.6693	+0.6663	+1.7813	
	11 SIDE RAIL	-0.6099	+0.5635	+0.9318	
_	6 HEAD REST	+0.0230	+0.5531	+2.7161	
-5 ⁰	8 LOWER PLATE	-0.6726	+0.6689	+1.7729	
	11 SIDE RAIL	-0.4423	+0.5635	+1.0214	
	6 HEAD REST	+0.2618	+0.5531	+2.7036	
co	8 LOWER PLATE	-0.6654	+0.6689	+1.7969	
	11 SIDE RAIL	-0.3580	+0.5635	+1.0531	

dimensions are referenced to the seat reference point (SRP). The seat reference point is located at the inter-section of the horizontal seat plate (x axis) center line and the vertical back plate (z axis) center line.

FIGURE A-10: FIDUCIAL TARGET LOCATIONS







.















VSVDØC PAGE 2 OF 2





FIGURE A-15: PROGRAM FLOWCHART FOR VSVDØD

VSVDØD

PAGE 1 OF 1



ADDENDUM

TEST CONFIGURATION AND

DATA ACQUISITION SYSTEM FOR THE EFFECTS OF SEAT CUSHIONS AND SEAT BACK

ANGLE ON HUMAN RESPONSE DURING +Gz

IMPACT ACCELERATION

TEST PROGRAM

PHASE II

Prepared under Contract F33615-86-C-0531

> Prepared by Marshall Z. Miller

DynCorp (formerly Dynalectron Corporation) AAMRL Division Building 824, Area B Wright-Patterson AFB, Ohio 45433

June 1987

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INTRODUCTION

This report was prepared by DynCorp (formerly Dynalectron Corporation) for the Harry G. Armstrong Aerospace Medical Research Laboratory (AAMRL/BBP) under Air Force Contract F33615-86-C-0531.

The information provided herein describes all of the instrumentation, calibration, seat cushion and processing program changes of Phase II (when compared to Phase I) of the Effects of Seat Cushions and Seat Back Angle on Human Response During +Gz Impact Acceleration Test Program. Where identical requirements exist for both Phases I and II of the test program, these requirements will not be described herein. One hundred forty-five tests were conducted for Phase II during November and December 1986 and January 1987 on the Vertical Deceleration Tower test facility.

This report is to be used as an addendum to the DynCorp November 1986 test report for Phase I of the Effects of Seat Cushions and Seat Back Angle on Human Response During +Gz Impact Acceleration Test Program.

1. INSTRUMENTATION CHANGES

The instrumentation changes as required by the Phase II Test Plan will be detailed in this section if they were different than required by the Phase I Test Plan. Identical instrumentation as required by both test plans will not be discussed.

Tables A-3a through A-3c list all of the measurement instrumentation used in Phase II of the test program. These tables designate the manufacturer, type, serial number, sensitivity and other pertinent data on each transducer used. Table A-4 lists the manufacturers' typical transducer specifications.

Human head accelerations were measured using three Endevco Model 7264-200 linear accelerometers and one Endevco Model 7302A angular (Ry) accelerometer in Phase II. The Endevco Model 2264-200 linear accelerometers used in Phase I are more prone to temperature drift and offset caused by wiring deflection. The angular accelerometer was not changed. Figure A-17 illustrates the human head acceleration package.

Human thoraxic accelerations measured in Phase II of the test program was not required to be measured in Phase I. Human thoraxic accelerations were measured using two Entran Model EGAXT-100 linear accelerometers for x and z accelerations and one Entran Model EGAXT-250 linear accelerometer for y accelerations. The accelerometers were mounted on a one inch diameter by 1/8 inch thick acrylic plastic pad and were attached to the subjects' intervertebral space between T-4 and T-5 with double-backed tape. Figure A-18 illustrates the human thoraxic acceleration package.

2. SEAT CUSHIONS

The Aces II seat cushion was the only one tested during Phase I of the test program. Figure A-19 illustrates the Aces II seat cushion installed on the VIP seat fixture.

Phase II of the test program included testing the Aces II, Operational F-4 and the Confor Foam F-4 seat cushions.

The Operational F-4 seat cushion uses a double layered contoured foam cushion with a contoured survival kit lid. Figures A-20 and A-21 illustrate the Operational F-4 seat cushion.

The Confor Foam F-4 seat cushion uses a single layer flat foam cushion with a flat survival kit lid. Figures A-22 and A-23 illustrate the Confor Foam F-4 seat cushion.

3. CALIBRATION

Calibrations were performed before and after testing to confirm the accuracy and functional characteristics of the transducers. Pre-program and post-program calibrations for Phase II of the test program are given in Tables A-5a through A-5d.

4. PROCESSING PROGRAMS

The Fortran processing programs that were developed to process the test data for Phase I of the test program are called "VSVDOA," "VSVDOB," "VSVDOC," "VSVDOD," and "VSVDOE." These processing programs were modified for Phase II of the test program to include current transducer sensitivities and the human thorax x, y, z and resultant accelerations.

	CORPORATION	SPECIAL MOTATIONS															- 28.25 lb - 29.75 lb - 27.5 l <mark>b page 1 Of 3</mark>
	ZO	BRIDGE COMPLETION REUSTONS	1	1.58K	1.5K	1.5K	1.5K	1.5K	1.65K	1.65K	1.65K	1	ı	1	,	1	SEAT WEICHT AT WEICHT AT WEICHT
	LECTA	BALANCE RALANCE RESISTORS	375K -in Gd.	43K +in Gd.		'	. '	1	1.2M -in Gd.	-	0-in Gd.	۰ د م م		ہ ر	، حق	0 - in cd.	CELL A-H Cell J Se Cell J Se
	IVNAC	XDUCEN ZERO AANGE	2.5	2:5 0:0 0.0	2.5	2.5	2.5 •5.6	2.5	2.5 •5.(2.5 +5.	2.5	2.5	2. 2. 2.0	<u>;</u> <u>;</u> ;;;	2. 2. 2.0	2. 2. 2. d	
	н	FILTER NZ	120	120	120	120	120	120	120	120	120	120	120	120	120	120	
	13M 87	F.S. SENS	23.8 G	9.06 C	9.0 C	19.3 G	17.8 G	40.1 G	17.9 C	10.3 C	38.0 C	1551 1b	1550-16	3094 Ib	603 Ib	582 lb	
	<u>- 51 - UNN</u>	SAMPLE RITTE FORMAT	т Т	× -	IK I	1K 1	1K _ 1	1K 1	IK I	1 IK	I H	1 IK	- K	- 	ž	× ×	
ENTS	04 86	A MAN	25 29	100	100 4	50 23	20	25	50 2	100 37	25 34	201	201	3	402	402 5	
EQUIREM		FILTEN EALES S/M	1 0,9	60 2	- 0,9	09	09	° ° °	, 09	809	609	60 10	60 11	60 12	60 13	60 14	
ATION R	BACK E 11)	EACITE V CHAN	10.00	2 00.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	10.00	-
TRUMENT	AND SEAT	xbucen sens	4.196 #v/G	2.759 ••/G	2.783 #v/G	2.592 #v/G	2.814 #v/C	2.492 #v/G	2.795 mv/G	2.435 ≣v/G	2.629 #v/G	8.02 uv/1b	8.03 uv/1b	8.08 uv/1b	10.32 uv/1b	10.68 uv/1b	
TAL INS	T CUSHION ACCELERATI	N/8	FR42	BX17	вн97н	BH58H	BH60H	внбЭн	BC26	BB 13	2A20	r 3294-1	r 3294-2	I 3294-4	2	n	
DIG	CTS OF SEA URING +GZ DECELERAT	XDUCER MFG A TYPE	ENDEVCO 2262A-200	ENDEVC0 2264-200	ENDEVC0 7264-200	ENDEVC0 7264-200	ENDEVC0 7264-200	ENDEVCO 7264-200	ENDEVC0 2264-150	ENDEVCO 2264-150	ENDEVCO 2264-150	STRAINSEF FL2.5u- 2SPKT	STRAINSEF FL2.5u- 2SPKT	STRAINSEF FL2.5u- 2SPKT	MM/DYN EA06-062 TJ-350	MM/DYN EA06-062 TJ-350	
	THE EFFE ANCLES D VERTICAL	DATA POINT	CARR LAGE Z	CARR LAGE X	CARR LAGE Y	HEAD X	HEAD Y	HEAD Z	CHEST X	CHEST Y	CHEST Z	LEFT SEAT LOAD	RIGHT SEAT LOAD	CENTER SEAT LOAD	LEFT LOAD LINK X	R ICHT LOAD LINK X	
	FROGRAM FACILITY	DATA CHANNEL	-	2		4	2	9	~	80	6	10	11	12	13	14	

TABLE A-3a: INSTRUMENTATION REQUIREMENTS

•

		DIG	ITAL INS	STRIMEN	TATION		MENTO								Γ
PROGRAA FACILITY_	A THE EP ANGLES VERTIC	FECTS OF S DURING +G	EAT CUSHIC 2 ACCELERA	N AND SEA VTION (PHA	T BACK SE II)		NOV 86	<u>SL UNT.</u>	JAN 87	н	INNY	LECTI	Nox Nox	CORPORATION	
DATA CHANNEL	DATA POINT	XDUCEN MFG A	¥/8	XDUCER SEMS	EXCITE	FILTER		SAMPLLE SAMPLLE	F.B. SERS	FILTER MZ	XDUCEA ZENO AANDE	BRIDGE BALANCE NEBHETONE	BRIDGE COMPLETION RESISTONS	SPECIAL NOTATIONS	
15	LEPT LAP	GH 3D- SW	15X	5.35 uv/1b	10.00	° °	402	×	1162 lb	120	2.5	16K +in Cd.	 		
16	LEFT LAP LOAD Y	GH- 3D- SW	15Y	5.32 uv/1b	10.00	9 9	800	×	587 Ib	120	2.5	18K +in Gd.	,]
11	LEFT LAP LOAD Z	CH- 3DSW	152	6.30 uv/1b	10.00	\$ \$	402	×	987 Ib	120	2.5	5K +in Gd.	•		
18	R IGHT LAP LOAD X	GM- 3D-SW	21X	5.04 uv/1b	10.00	09	402		d1 4621	120	2.5	69K -in Gd.			
19	R IGHT LAP LOAD Y	c H 3b-SW	21Y	4.83 uv/1b	10.00	09 09	800	, T	647 lb	120	2.5 5.0	17K +in Cd.	1		
20	R IGHT LAP LOAD Z	CH- 3D-SU	212	6.09 uv/1b	20	60 20	402	IK I	1021 lb	120	2.5	27K -in Gd.	,		
21	SHOULDER LOAD X	GM- 3D-SW	202	6.30 uv/1b	21	50	402	- ×	987 1b	120	2.5	46K - in Gd.	 '		
22	SHOULDER LOAD Y	GM- 3D-SW	20Y	5.81 uv/1b	10.00	60 22	800	×	538 lb	120	2.5 45.0	178K +in Gd.			
23	SHOULDER LOAD Z	CH- 3D-SW	2 OX	5.58 uv/1b	10.00	23	402	× -	111516	120	2.5 45.0	29K +in Gd.	1		
25	SEAT X ACCEL.	ENDEVCO 2264-150	BB28	2.700 ■v/c	10.00	60 25	05 L	- ¥	18.5 C	120	2.5 \$5.0	72.4K +in Gd.	1.65K		
26	SEAT Y ACCEL.	ENDEVCO 2264-200	BV 95	2.981 mv/G	10.00	60 26	50 21	 ¥	16.8 C	120	2.5 5.0	372K +in Gd.	1.47K		
27	SEAT Z Accel.	ENDEVCO 2264-200	BW07	2.824 #v/c	10.00	09 27	<u>я</u>	 ¥	17.7 C	120	2.5	294K -in Gd.	1.47K		
28	CENTER LOAD LINK Y	ЮН/ DYN EA-06-062 TJ-350	\$	9.82 uv/1b	28	60 28	402	 =	633 1b	120	2.5				Τ
29	VEL. TACH.	CLOBE 22A672-2	4	.06209 V/F/S	- 29	° 2	-	- ¥	80.5 FT/SEC	60	0.0	1			
														PAGE 2 OF 3	

TABLE A-3b: INSTRUMENTATION REQUIREMENTS

CORPORATION	SPECIAL NOTATIONS															PAGE 3 OF 3
LON OL	BRIDGE COMPLETION AEBISTONS	1,65K	1.65K	1.65K	1	1	ı	•	1	1	•	1	•	1	5	
RCTI	INIDOL BALANCE RESISTONS	190K +in Gd.	220K +in Gd.	80K +in Gd.	I	,	1	1	1	1	-	450K -in Gd.	'	'	1	
IVNI	XDUCER ZENO RANGE	2.5 +5.0	2.5	2.5 45.0	2.5.0	2.5	2.5 45.0	2.5 45.0	5.0 45.0	0.0	2.5	2.5	2.5	2.5	2.5 45.0	
н	FILTER HZ	120	120	120	120	120	120	120	2000	2000	120	120	120	360	360	
JAN 87	F.S. 36H5	17.0 G	18.2 C	37.4 G	3027 RAD/SEC ²	24.9 C	24.9 G	49.1 G	5.0 Volt	5.0 Volt	2961 RAD/SEC ²	1840 RAD/SEC ²	1666 RAD/SEC ²	2.5 Volt	2.5 Volt	
THR U 15	SAMPLE SAMPLE	1K 1	1K	1 1	11	IK I	IK I	× ×	IK I	- K	×	- IK	×-	л Т	1K _1	
IENTS 0V [.] 86		50 %	50 5	25 18	100	100 30	201	50 26	2.5	-	201	201	402	-	-	
EQUIREM DATE 21 N	FILTER SAN	41	60 31	60 32	60 33	46 03	60 35	60 36	1000	1600	60 42	60 43	60	180	180	
ATION R ACK 11)	V CHAN	0.00	16.00	10.00	10.09	34	10.00	36	- 18	- 38	10.00	10.00	10.00		- 48	
FRUMENT AND SEAT B ON (PHASE	xpuces see	2.939 #v/G	2.740 =v/G	2.676 #v/G	8.26 uv/BAD/ SEC	1.005 ■v/c	.500 ••/G	1.019 wv/G	1	ı	4.20 uv/BAD/ SEC	6.76 uv/ BA D/ SEC	3.732 uv/BAD/ BEC	ı	1	
TAL INS	¥,	CH74	BQ4.2	CH70	A150	12Q6S- L1-1	12q6 <mark>8-L</mark> 14-14	29T6U-A 18-18	1	ı	AB12	A B15	PT47	ŀ	1	
DIGI DIGI URING +CZ DECELERAT	XDUCEN MFG A TYPE	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 7302	ENTRAN ECAXT-100	ENTRAN EGAXT-250	ENTRAN BCAXT-100	1	1	ENDEVCO 7302A	ENDEVCO 7302A	ENDEVCO 7302B	1	1	
THE EFFE ANGLES DI VERTICAL	DATA POINT	DUMMY HEAD X	DUMMY HEAD Y	DUNMY HEAD Z	DUNNY HEAD ANG	THORAX X	THORAX	THORAX 2	EVENT	T=0 PULSE	HEAD RY ANGULAR	CHEST RY ANGULAR	SEAT RY ANGULAR	2.5 Volt Bias	10 Volt Exc.	
PROGRAM FACILITY	DATA Chammel	30	31	32	66	34	35	36	37	38	42	43	44	47	87	

TABLE A-3c: INSTRUMENTATION REQUIREMENTS

.

	SENSITIVITY RESONANCE FREQUENCY EXCITATION 2 ARM ADDITIONAL (mv) FREQ (Hz) RESPONSE (Hz.) (Volt) or 4 ARM NDTES	2.5/G 3400 0-800 10 2 am Linear accelerometer	2.5/G 4700 0-1200 10 2 am Linear accelerometer	2.5/G 6000 0-1000 10 2 arm Linear accelerometer 1000 G overrange	2.5/G 7000 0-1800 10 4 arm Linear accelerometer, .7 damping ratio	.006 2250 1–600 10 4 am Angular Accelerometer, /Rad/Sec2 X10 overrange; housing connector	.055 2500 1–600 10 4 arm Angular accelerometer, 2. /Rad/Sec2 X10 overrange	.004 3000 1–600 10 4 arm Angular accelerometer, 2. /Rad/Sec2 X10 overrange	2.0/G 1700 0-800 10 4 am Linear accelerometer; 10 KG overrange, .7 damping ratio	1.0/G 2000 0-1000 10 4 arm Linear accelerometer; 10 KG overrange, .7 damping ratio	.008/Lb 3600 0-2000 10 4 arm Load cell; 15 V max exc.;
	ENSITIVITY RESONANCE (mv) FREQ (Hz)	2.5/G 3400	2.5/G 4700	2.5/G 6000	2.5/G 7000	.006 2250 /Rad/Sec2	.055 2500 /Rad/Sec2	.004 3000 /Rad/Sec2	2.0/G 1700	1.0/G 2000	.008/Lb 3600
- 1	ANCE SENSITIVITY (mv)	150 G 2.5/G	200 G 2.5/G	200 G 2.5/G	200 G 2.5/G	50,000 .005 VSec2 /Rad/Sec2	0,000 .055 1d/Sec2 /Rad/Sec2	0,000 .004 id/Sec2 /Rad/Sec2	00 G 2.0/G	50 G 1.0/G	500 Lb .008/Lb
	R MODEL R	2264-150 ±1	2264-200 ±2	7264-200 ±2	2262 A- 200 ±2	7302 ±5 Rad	7302A ±5 Ra	73028 ±5 Ra	EGAXT-100 ±1	EGAVT-250 ±2	FL2.5U- ±21
	MANUFACTURE	Endevco	Endevco	Endevco	Endevco	Endevco	Endevco	Endevco	Entran	Entran	Strainsert

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TABLE A-4: TYPICAL TRANSDUCER SPECIFICATIONS

TYPICAL TRANSDUCER SPECIFICATIONS

į

PROGRAM: VSBA II

DECELER	
VERTICAL	
ACILITY:	

DATES: 21 NOV 86-15 JAN 87 RUN NUMBERS: 1215-1359 RATION

	TOWER							
	83.717637464		-38 <i>4</i>	CAL	-T204	CAL		
DATA POINT	MCG. L NODEL	NUNBER	DATE	SENS	DATE	SENS	A CHANGE	NOTES
CARRIAGE Z	ENDEVCO 2262A-200	FR42	98NON81	4.196	20JAN87	4.171	- e	
CARRIAGE X	ENDEVCO 2264-200	BX17	18NOV86	2.759	20JAN87	2.779	+.7	
CARRIAGE Y	ENDEVCO 7264-200	ВН97Н	18NOV86	2.783	20JAN87	2.781	-,1	
HEAD X	ENDEVCO 7264-200	вн58н	18NOV86	2.592	2 LJAN87	2.595	+.1	
HEAD Y	ENDEVCO 7264-200	внбон	18NOV86	2.814	2 1 J AN 8 7	2.803	4	
HEAD Z	ENDEVCO 7264-200	вн6 3н	18NOV86	2.492	21JAN87	2.492	0	
CHEST X	ENDEVCO 2264-150	BC 26	140CT86	2.795	2 1JAN87	2.792	1	
CHEST Y	ENDEVCO 2264-150	BE13	150CT86	2.435	2 1 J A N 8 7	2.443	+.3	
CHEST Z	ENDEVCO 2264-150	2A20	150CT86	2.629	21JAN87	2.664	+.2	
								All sensitivitie in units of mv/G

Page l of 4

TABLE A-5a: TRANSDUCER PRE- AND POST-CALIBRATION

PROGRAM: VSBA II

DATES: 21 NOV 86-15 JAN 87

F 6	ACILITY: VERTIC TOWER	AL DECE	<u>LERAT</u> ION	RUN	NUMBERS	1215-	1359	1
			-384	CAL	POST-	ĊAL	Γ	
DATA POINT	TRANSDUCEK NFG. & HODEL	NUMBER	DATE	SENS	DATE	SEMS	& CHANGE	MOTES
LEFT LOAD LINK X	MM/DYN EA-06-062TJ- 350	2	9870NET	10.32	15JAN87	10.35	+.3	
RIGHT LOAD LINK X	MM/DYN EA-06-062TJ- 350	3	13NOV86	10.68	15JAN87	10.66	2	
CENTER LOAD LINK Y	MM/DYN EA-06-062TJ- 350	5	13NOV86	9.82	15JAN87	9.86	+.4	
LEFT LAP LOAD X	CM-3D-SW	15X	13NOV86	5.35	16JAN87	5.36	+.2	
LEFT LAP LOAD Y	GN- 3D- SW	15Y	13NOV86	5.32	16JAN87	5.32	0	
LEFT LAP LOAD Z	G M- 3D-SW	152	1 3NOV 86	6.30	16JAN87	6.33	+.5	
RIGHT LAP LOAD X	GM- 3D- SW	2 I X	1 3NOV 8 6	5.04	16JAN87	5.04	0	
RIGHT LAP LOAD Y	GM- 3D-SW	21Y	13NOV86	4.83	16JAN87	4.80	6	
RIGHT LAP LOAD Z	MS-JD-SW	212	13NOV86	6.09	16JAN87	6.05	2	
SHOULDER LOAD X	GM- 3D-SW	20Z	14NOV86	6.30	16JAN87	6.29	2	All sensitivities in units of uv/Jb

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TABLE A-5b: TRANSDUCER PRE- AND POST-CALIBRATION

PROGRAM: VSBA II

DATES: 21 NOV 86-15 JAN 87

		NOTES							Unless noted otherwise, all	sens. in units of uv/lb.
6001		K CHANGE	-1.2	2	+.4	+.8	3			
	CAL	SENS	5.73	5.57	4.22 uv/RAD/ SEC	6.81 uv/RAD/ SEC	3.719 uv/RAQ/ SEC			
NUMBERS	POST	DATE	16JAN87	16JAN87	22JAN87	2 2 J A N B 7	22JAN87			
	CAL	SENS	5.81	5.58	4.20 JV/RAD/ SEC	6.76 JV/RAD/ SEC	3.732 1v/RAD/ SEC			
T.I.A	-3#d	DATE	14NOV86	14NOV86	19NOV86	19NOV86	2 2AUG86			
LCAL DEC	4501A1	NUMBER	20Y	20X	AB12	AB15	PT47			
CILITY: VERT TOWE	TPANCNICED	MEG. & HODEL	GM- 3D- SW	GM- 3D- SW	ENDEVCO 7302A	ENDEVCO 7302A	ENDEVCO 7302B			
FA		DATA POINT	SHOULDER LOAD Y	SHOULDER LOAD Z	HEAD RY ANGULAR	CHEST RY ANGULAR	SEAT RY ANGULAR			

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TABLE A-5c: TRANSDUCER PRE- AND POST-CALIBRATION

PROGRAM: VSBA II

DATES: 21 NOV 86-15 JAN 87

FACILITY: VERTICAL DECELERATION TOWER

RUN NUMBERS: 1215-1359

		NOTES								Unless noted otherwise. all	sens. in units of mv/G.
		K CHANGE	+.5	1	4.4	+.1	1	+.5	-1.2		
	r-cal	SENS	2.714	2.979	2.835	2.943	2.736	2.690	8.154 uv/BAD/ SEC		
	SOd	DATE	20JAN87	20JAN87	20JAN87	21JAN87	2 1 J A N B 7	21JAN87	22JAN87		
	-CAL	SENS	2.700	2.981	2.824	2.939	2.740	2.676	8.257 uv/BAD/ SEC		
	344	DATE	18NOV86	18NOV86	18NOV86	22AUG86	2 2 AUG 8 6	22AUG86	19NOV86		
	SERIAL	NUMBER	BB28	BV95	BW0 7	CH74	BQ42	CH70	A150		
T T T T	TRANSDUCER	MFG. L NODEL	ENDEVCO 2264-150	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 2264-200	ENDEVCO 7302		
		DATA POINT	SEAT X ACCELERATION	SEAT Y ACCELERATION	SEAT Z ACCELERATION	DUMMY IIEAD X	DUMMY HEAD Y	DUMMY HEAD Z	DUMMY HEAD ANGULAR		

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TABLE A-5d: TRANSDUCER PRE- AND POST-CALIBRATION


FIGURE A-17: HUMAN HEAD ACCELERATION PACKAGE



FIGURE A-18: HUMAN THORAXIC ACCELERATION PACKAGE



FIGURE A-19: ACES II SEAT CUSHION



FIGURE A-20: OPERATIONAL F-4 SEAT CUSHION



FIGURE A-21: OPERATIONAL F-4 SEAT CUSHION



FIGURE A-22; CONFOR FOAM F-4 SEAT CUSHION



FIGURE A-23: CONFOR FOAM F-4 SEAT CUSHION

APPENDIX B

REPRESENTATIVE TEST DATA

DATA ID	IMMEDIATE Preimpact	MAXIMUM VALUE	MINIMUM VALUE	TIME OF MAXIMUM	TIME OF MINIMUM
REFERENCE MARK 2.5V EXT PWR 10V EXT PWR		2.50 10.01	2.50 10.00	-150. 0. 1.	10. 0.
CARRIAGE ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS X AXIS (SM)	0.07 -0.03 0.05 0.07	0.53 1.44 9.86 9.65	-0.49 -1.61 0.50 0.56	44. 57. 67. 69.	23. 52. 0. 0.
SEAT ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS Z AXIS (SM) RY	0.05 -0.02 -0.24 -0.24 -0.24 -8.26	1.69 0.91 10.72 9.89 26.72	-1.74 -0.77 0.03 0.11 -33.25	56. 79. 61. 71. 61.	52. 8. 0. 65.
CARRIAGE VELOCITY (F/S)	-26.81	-1.24	-27.40	333.	9.
CHEST ACCELERATION (G) X AXIS Y AXIS Z AXIS RESULTANT NORM RESULTANT SI RY	-0.06 -0.74 -0.83 1.11 0.12 5.69	1,25 0.86 14.81 14.88 1.54 27,63 219.60	-0.78 -1.08 -0.77 0.59 0.06 -304.57	76. 78. 76. 76. 76.	107. 155. 3. 170. 170. 92.
HEAD ACCELERATION (G) X AXIS Y AXIS Z AXIS RESULTANT NORM RESULTANT SI RY	-0.35 -0.48 -0.59 0.82 0.09 0.69	1.95 0.19 13.60 13.66 1.42 22.19 161.45	-3.29 -1.44 -1.33 0.34 0.04 -134.57	157. 169. 71. 71. 71. 73.	109. 67. 165. 178. 178. 164.
THORAX ACCELERATION (G) X AXIS Y AXIS Z AXIS Resultant Norm resultant	-0.59 0.17 -0.31 0.70 0.07	0.62 0.89 14.02 15.14 1.57	-6.91 -3.13 -0.45 0.08 0.01	162. 161. 88. 88. 88.	78. 81. 164. 167. 167.
SHOULDER LOADS (LB) X AXIS Y AXIS Z AXIS RESULTANT	87.28 -2.46 2.10 87.35	125.73 7.69 38.39 129.74	33.96 -3.06 1.63 34.83	61. 215. 85. 61.	301. 4. 1. 301.
LAP LOADS (LB) LEFT X AXIS LEFT Y AXIS LEFT Z AXIS LEFT RESULTANT RIGHT X AXIS RIGHT Y AXIS RIGHT Z AXIS RIGHT RESULTANT	67.79 24.48 59.48 93.45 54.99 19.89 61.81 85.09	87.01 26.40 56.56 103.13 23.35 23.35 59.10 99.01	17.29 8.78 -1.66 23.43 11.25 3.30 -0.10 16.12	98. 96. 98. 98. 99. 99. 97.	338. 52. 60. 362. 357. 61. 57.
SEAT LOADS (LB) LEFT LINK X AXIS RIGHT LINK X AXIS X AXIS CENTER LINK Y AXIS LEFT PAN Z AXIS RIGHT PAN Z AXIS CENTER PAN Z AXIS Z AXIS SUM Z AXIS MINUS TARE RESULTANT RESULTANT MINUS TARE	$\begin{array}{c} -0.87\\ 2.08\\ 1.21\\ -2.50\\ 46.76\\ 37.75\\ 100.21\\ 184.72\\ 219.70\\ 184.75\\ 219.72\\ 219.72\end{array}$	$\begin{array}{c} 1.84\\ 8.08\\ 8.16\\ -1.38\\ 624.40\\ 594.40\\ 1437.98\\ 2565.48\\ 2337.99\\ 2565.76\\ 2338.30\end{array}$	-12.01 -26.25.88 -35.88 -36.83 36.86 30.84 114.25 189.71 196.25 189.71 196.25	164. 46. 274. 84. 84. 78. 78. 78. 78. 78.	82. 92. 93. 72. 360. 325. 320. 320. 322. 320. 322.

VSBR STUDY II TEST: 1295 SUBJ: D-5 WT: 172.0 NOM G: 10.0 CELL: F











450 300 SEAT RY (RAD/S##2) 150 0 SUBJ ID: D-5 TIME IN MILLISECONDS -150 1000.0 1 -500.0 --1000.0 -500.0 -0.0 1295 450 1000.0 7 CHEST RY (RAD/S**2) 300 TEST NO: 1000.0 T HEAD RY (RAD/SXX2) 150 **USBA STUDY II** 0 -150 -500.0 --1000.0 --500.0 -500.0 -0.0 500.0 --1000.0 -0.0













VSBA STUDY II TEST: 1333 SUBJ: 0-5 WT: 172.0 NOM G: 10.0 CELL: J

DATA ID	IMMEDIATE PREIMPACT	MAXIMUM VALUE	MINIMUM VALUE	TIME OF MAXIMUM	TIME OF MINIMUM
REFERENCE MARK 2.5V EXT PHR 10V EXT PHR		2.50 10.01	2.50 10.00	-147. 1. 0.	0.
CARRIAGE ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS Z AXIS (SM)	-0.07 0.03 0.07 0.08	0.57 1.60 9.96 9.73	-0.48 -1.59 0.50 0.52	47. 58. 69. 69.	166. 54. 0. 0.
SEAT ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS Z AXIS (SM) RY	-0.01 -0.07 0.14 0.14 3.43	1.79 0.79 11.16 10.28 36.75	-1.87 -0.75 0.33 0.41 -23.22	57. 80. 62. 71. 62.	53. 30. 0. 57.
CARRIAGE VELOCITY (F/S)	-26.83	-1.20	-27.73	357.	14.
CHEST ACCELERATION (G) X AXIS Y AXIS Z AXIS RESULTANT NORM RESULTANT SI RY	0.10 -0.90 -0.86 1.25 0.13 6.09	4.47 2.61 15.33 16.10 1.65 29.50 156.40	-0.31 -1.24 -0.91 0.69 0.07 -314.43	77. 86. 76. 76. 76. 80.	110. 27. 6. 17. 17. 91.
HERD ACCELERATION (G) X AXIS Y AXIS Z AXIS Resultant Norm Resultant SI RY	-0.25 -0.34 -0.47 0.63 0.07 -6.44	1.24 0.81 15.20 15.26 1.57 24.92 281.04	-3.50 -0.87 -0.65 0.21 0.02 -204.43	72. 198. 74. 74. 74. 74. 76.	116. 67. 1. 369. 369. 117.
THURAX ACCELERATIUN (G) X AXIS Y AXIS Z AXIS Resultant NORM Resultant	-0.17 0.99 -0.31 1.05 0.11	0.44 1.62 21.51 21.81 2.24	-6.94 -0.65 -0.66 0.89 0.09	154. 158. 83. 83. 83.	96. 81. 158. 12. 12.
SHOULDER LOADS (LB) X AXIS Y AXIS Z AXIS RESULTANT	61.29 2.76 -3.31 61.45	119.39 8.12 34.31 123.58	28.61 -1.02 -3.57 28.76	101. 97. 92. 101.	377. 34. 1. 382.
LAP LOADS (LB) LEFT X AXIS LEFT Y AXIS LEFT Z AXIS LEFT RESULTANT RIGHT X AXIS RIGHT Y AXIS RIGHT Z AXIS RIGHT RESULTANT	45.05 13.73 41.27 62.63 26.31 11.76 31.99 43.06	76.78 17.03 45.29 90.27 66.06 25.49 47.81 83.93	15.20 4.70 -0.10 18.06 5.62 1.57 -3.23 7.81	96. 95. 1. 99. 99. 99. 100. 97.	200. 212. 53. 200. 198. 201. 50. 198.
SEAT LUADS (LB) LEFT LINK X AXIS RIGHT LINK X AXIS X AXIS CENTER LINK Y AXIS LEFT PAN Z AXIS RIGHT PAN Z AXIS CENTER PAN Z AXIS Z AXIS SUM Z AXIS MINUS TARE RESULTANT RESULTANT MINUS TARE	$\begin{array}{r} 0.74 \\ -3.63 \\ -2.80 \\ -7.00 \\ 24.11 \\ 23.22 \\ 55.86 \\ 102.89 \\ 103.16 \\ 126.63 \end{array}$	16.94 6.16 13.18 -4.42 764.17 596.03 1421.95 2639.87 2402.29 2645.72 2408.69	-55.95 -11.89 -66.09 -170.27 30.91 24.71 64.21 122.92 140.45 123.49 140.94	178. 58. 177. 316. 83. 85. 72. 83. 77. 76. 77.	83. 85. 71. 0. 338. 1. 1. 1. 1.

























DATA ID	IMMEDIATE Preinpact	MAXIMUM VALUE	MINIMUM Value	TIME OF MAXIMUM	TIME OF MINIMUM
REFERENCE MARK 2.5V EXT PWR 10V EXT PWR		2.50 10.00	2.50 9.99	-148. 21. 2.	0: 0:
CARRIAGE ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS Z AXIS (SM)	0.00 -0.07 0.05 0.06	0.65 2.82 10.02 9.74	-0.67 -1.77 0.49 0.53	44. 12. 67. 68.	16. 52. 0. 0.
SEAT ACCELERATION (G) X AXIS Y AXIS Z AXIS Z AXIS Z AXIS (SM) RY	-0.05 0.01 0.07 0.07 -1.41	2.41 1.52 10.85 10.26 23.39	-1.85 -1.04 0.37 0.37 -31.58	11. 50. 68. 69. 44.	52. 70. 0. 21.
CARRIAGE VELOCITY (F/S)	-27.01	-1.21	-27.58	358.	6.
CHEST ACCELERATION (G) X AXIS Y AXIS Z AXIS RESULTANT NORM RESULTANT SI RY	0.14 -0.68 -0.74 1.02 0.10 -4.49	3.95 -0.08 17.07 17.20 1.77 \$5.81 358.68	-1.04 -1.78 -0.80 0.59 0.06 -382.51	88. 95. 75. 76. 76. 77.	110. 57. 7. 14. 14. 92.
HEAD ACCELERATION (G) X AXIS Y AXIS Z AXIS RESULTANT NORM RESULTANT SI RY	-0.28 -0.35 -0.49 0.67 0.07 -6.88	0.81 0.22 16.94 16.98 1.74 28.47 580.38	-3.54 -1.19 -0.59 0.38 0.04 -239.60	73. 131. 74. 74. 74. 73.	118. 68. 10. 344. 344. 119.
THORAX ACCELERATION (G) X AXIS Y AXIS Z AXIS Resultant Norm resultant	-0.30 0.45 -0.29 0.62 0.06	0.30 5.79 20.57 20.86 2.14	-6.76 -0.35 -0.37 0.28 0.03	189. 85. 92. 92. 92.	86. 22. 167. 171. 171.
SHOULDER LOADS (LB) X AXIS Y AXIS Z AXIS RESULTANT	75.17 -0.33 -7.93 75.59	116.57 12.03 33.24 119.83	29.74 -4.10 -7.98 29.77	104. 69. 91. 104.	422. 147. 0. 450.
LAP LOADS (LB) LEFT X AXIS LEFT Y AXIS LEFT Z AXIS LEFT RESULTANT RIGHT X AXIS RIGHT Y AXIS RIGHT Z AXIS RIGHT RESULTANT	41.21 10.87 43.84 61.15 39.53 16.05 47.35 63.73	56.40 12.30 45.13 68.70 43.47 16.57 48.45 65.63	2.95 -2.97 -13.08 5.22 0.30 -0,25 -12.80 8.26	107. 9. 108. 103. 1. 0. 3.	56. 59. 59. 57. 59. 56. 48.
SEAT LOADS (LB) LEFT LINK X AXIS RIGHT LINK X AXIS X AXIS CENTER LINK Y AXIS LEFT PAN Z AXIS RIGHT PAN Z AXIS CENTER PAN Z AXIS Z AXIS SUM Z AXIS MINUS TARE RESULTANT RESULTANT MINUS TARE	-2.75 -3.25 -6.00 11.70 21.45 30.24 85.25 136.93 164.69 197.56 165.22	19.75 30.39 41.11 15.18 744.45 600.37 1639.63 2696.46 2445.07 2696.52 2445.13	$\begin{array}{c} -10.37\\ -8.02\\ -14.20\\ -19.64\\ 18.94\\ 25.93.22\\ 144.30\\ 153.42\\ 145.10\\ 153.57\end{array}$	12. 98. 952. 94. 94. 82. 83. 83. 83. 83.	81. 133. 69. 4. 353. 0. 9. 9. 9.

VSBR STUDY II TEST: 1346 SUBJ: D-5 WT: 172.0 NOM G: 10.0 CELL: I










460 310 1000.0 7 SEAT RY (RAD/S##2) 160 10 SUBJ ID: D-5 TIME IN MILLISECONDS -140 -500.0 --1000.0-500.0 0.0 1346 460 1000.0 1 CHEST RY (RAD/S##2) 310 TEST NO: 1000.0 7 HEAD RY (RAD/S##2) 160 USBA STUDY II 10 -140 -1000.0 -500.0 --1000.0 --500.0 -0.0-500.0 -0.0 -500.0











