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MTL TR 90-42

AIRCRAFT QUALITY HIGH TEMPERATURE VACUUM CARBURIZING

November 1990

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FINAL REPORT

Contract DAAG46-82-C-0034

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Prepared for

U.S. ARMY MATERIALS TECHNOLOGY LABORATORY Watertown, Massachusetts 02172-0001

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Unclassified SECURITY CLASSIFICATION OF THIS PAGE

REPORT	DOCUMENTATIO	N PAGE			Form Approved OMB No. 0704-0188
To REPORT SECURITY CLASSIFICATION Unclassified		TH RESTRICTIVE	MARKINGS		
28 SECURITY CLASSIFICATION AUTHORITY		3 DISTRIBUTION	AVALABILITY	OF REPORT	
26 DECLASSIFICATION / DOWNGRADING SCHEDU	€	Approval unlimited	for public 1.	release	; distribution
4 PERFORMING ORGANIZATION REPORT NUMBE	ER(S)	S MONITORING ORGANIZATION REPORT NUMBER(S)			
ł		MTL TR 90-42			
6. NAME OF PERFORMING ORGANIZATION	60 OFFICE SYMBOL	74 NAME OF M	ONITORING ORG	ANIZATION	
Boeing Helicopters	(<i>IT applicable</i>) P38-21	U.S. Army M	laterials T	echnolog	jy Laboratory
6c ADDRESS (City, State, and ZIP Code)		76 ADDRESS (CH	ty, State, and Zil	Code)	
Philadelphia, PA 19142			nal St		
Dept. 8-7525		Watertown, MA 02172-0001			
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U.S. Army Mat'ls Tech. Lab.	SLCMT-EMM	DAAG646	-82-C-0034		
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Metals Research Branch		CLEMENT NO	PRÓJECT	TASK	WORK UNIT
Watertown. MA 02172					
11 TITLE (Include Security Classification)					
Aircraft Quality High Tem	perature Vacuum	Carburizing	(Unclassif	ied)	
12 PERSONAL AUTHOR(S)					
R. J. Cunningham and R. J.	Drago			-	
TTO TYPE OF REPORT 136 TIME COVERED 14 DATE OF REPORT (Year Month, Day) 15 PAGE COUNT Final FROM <u>4/82 TO 2/90</u> 1990, NOVEMBER 211					
16 SUPPLEMENTARY NOTATION					
	C -				
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	Scoring, S	ingle Tooth	Bending Fat	tique, S	urface Dur-
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a spiral bevel input pinion gear was produced, vacuum carburized, and then tested in an					
actual helicopter transmission. Metallurgical evaluation of the tested gear showed that it					
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Although a procedure for Varco X2M was developed in the Varco X2M alloy.					
for this alloy is presented. <					
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Martin Wells		(617)923-53	27	SLC	MT-EMM
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SUMMARY

Development of a high temperature vacuum carburizing procedure for AISI 9310 gear steel was accomplished during this program. Use of this procedure significantly reduced processing time of gears, which can lead to a reduction in acquisition costs. The procedure was utilized to process surface contact fatigue, single-tooth bending fatigue, scoring, spur, and spiral bevel test gears. The data obtained from the testing and evaluation of these components was equal to or greater than similar data from conventionally carburized material. No significant variation in the vacuum carburize test data was observed from two mill heats of material. Following evaluation of these test gears, a spiral bevel input pinion gear was produced, vacuum carburized, and then tested in an actual helicopter transmission. Metallurgical evaluation of the tested gear showed that it met the same performance standards required of a conventionally carburized gear.

In addition, a vacuum carburization procedure was investigated for the Vasco X2M alloy. Although a procedure for Vasco X2M was developed, it was not optimized. The data generated for this alloy is presented.

PREFACE

This final contract report covers the work performed under contract DAAG646-82-C-G034 from April, 1982 to February, 1990 by Boeing Helicopters. The program was administered under the technical direction of Dr. Paul Fopiano (1982-1988) and Dr. Martin Wells (1988-1990), U.S. Army Materials Technology Laboratory, Metals Research Branch, Watertown, MA 02172.

The work was performed by Boeing Helicopters Materials Engineering Department. Phase I of this program was conducted under the technical direction of Mr. Gayle B. Wadsworth, former Manager of the Materials Engineering Department. Phases II, III, and IV were conducted under the technical direction of Mr. Roy J. Cunningham, Chief Metallurgist, Metallic Materials Engineering Department, and Mr. Ray Drago, Staff Engineer, Dynamic Systems Technology Department.

The major subcontractors for the program were C.I. Hayes, Inc., Cranston, RI. and Summit Jear Corporation, Plymouth, MN.

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ACKNOWLEDGEMENTS

Acknowledgement is made to Mr. Stan Casper of C.I. Hayes for his cooperation and assistance in developing the vacuum carburizing procedures, as well as heat treatment, of certain test specimens.

Acknowledgement is also made to Messrs. Maurice Taylor and Kenneth Anderson of Summit Gear Corporation for their cooperation and assistance in the timely and efficient machining and heat treatment of the test specimens.

Acknowledgement is also made to the following people who participated in gathering and disseminating the data contained in this report.

Mr. B. Boutilier Mr. E. Slate Mr. J. Kachelries Mr. M. Brody Mr. B. Johnson Mr. G. Turk Ms. P. Conyers

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1. INTRODUCTION

Carburization is one of the most costly and time consuming operations required in the manufacture of critical helicopter transmission components. The process provides a gear with a hardened tooth surface (after the teeth are cut) by the diffusion of carbon into the steel. After proper hardening, surface hardnesses of Rockwell C (R/C) 59-64 are obtained. The hard, carburized surface imparts excellent wear resistance properties as well as increases the load carrying capability of the component. Depending on the steel utilized, a degree of scoring and scuffing resistance is also attained. However, conventional endothermic carburizing processes require not only long process times of up to 35 hours (dependent upon the desired effective case cepth*), but significant amounts of energy input in the form of both electricity and natural gas. Special masking techniques and atmosphere controls are also necessary.

Vacuum carburizing, which is a relatively new (1970's), advanced carburizing process, has been recognized as an energy and cost effective alternative to conventional endothermic carburization. It has permitted the use of higher carburization temperatures, which can reduce carburization cycle times from as great as two weeks to less than one day. Energy input can be reduced in proportion. Furthermore, gaseous atmospheres are not needed for subsequent hardening operations.

Despite these advantages the aerospace industry, in general, has not been sufficiently convinced that vacuum carburization could be applied to critical aircraft components. However, development of a high temperature technique in recent years has altered this thinking. Primarily through the efforts of the C.I. Hayes Company in Cranston, RI, a viable process has been developed which has been applied to commercial components. However, qualification of the process for aerospace applications by testing specimens and helicopter transmission gears had not yet been accomplished until this program was completed.

*Effective case depth is defined as the perpendicular distance from the surface to a depth below the gear tooth surface at which R/C 50 occurs.

Boeing Helicopters took a special interest in this program because of its use of advanced, state-of-the-art carburization and heat treating practices for processing high hot hardness alloys in its advanced helicopter transmissions. Boeing Helicopters has utilized the experience gained from such heat treating practices to implement the use of the advanced high hot hardness Vasco X2M material in the CH-47 D & E model helicopter transmissions. Boeing developed the heat treat procedures for carburizing X2M as well as numerous other high hot hardness alloy steels. In addition, they have been involved in the development of vacuum heat treating since its conception in the early 1970's, and have been highly interested in the advantages of this type of processing over the much slower endothermic process. In-house research and development (IRAD) programs showed the feasibility of vacuum carburizing both the 9310 and Vasco X2M alloys. Data obtained under these IRAD programs on several test slugs from various vacuum carburizing runs showed that uniform carburization may be obtained, and cycle times and energy usage could be decreased.

To make effective use of vacuum carburizing in processing aerospace gear materials, the process had to be qualified by evaluating various vacuum carburized gears, as was done in this program. Successful development of the vacuum carburizing procedure can reduce gear processing time, energy consumption, and the cost of Army helicopter transmissions.

Successful completion of this program led to the following:

- (a) Generation of a high temperature vacuum carburizing procedure for AISI 9310 and Visco X2M gear steels
- (b) Data to qualify the process for aerospace components made from AISI 9310
- (c) An implementation plan for deploying the vacuum carburization process into the production of helicopter transmission components.
 - 2

2. OBJECTIVE

The objective of this program was to develop a high temperature vacuum carburizing technique for two gear materials which are presently being used in helicopter transmissions. Development, qualification, and incorporation of these procedures into production processing will result in significant cost savings through the use of higher carburization and heat treat temperatures that will shorten production process cycles. Accomplishing carburization and heat treatment under vacuum conditions will eliminate the need for pre-heating the part prior to heat treatment (necessary for high hot hardness gear steels) and will yield an energy savings due to lower usage of atmospheric gases.

3. SUMMARY OF TECHNICAL APPROACH

The program developed for the qualification of high temperature vacuum carburizing is graphically presented in Figure 1. This program was initially designed to be conducted in three phases. In Phase I, two heats of the AISI 9310 and VASCO X2M steel alloys were obtained, the vacuum carburization procedures were developed utilizing two vendors, and gear element testing was conducted. In Phase II, spur and bevel gears were produced and vacuum carburized using the procedures optimized in Phase I. The surface durability of these gears was then evaluated. In Phase III, a vacuum carburized input spiral bevel transmission gear was tested in a Boeing Helicopters CH-47C combining transmission, and a plan to implement the optimized vacuum carburization procedure into production was developed. However, during the course of the program, a fourth Phase was added in which further development of the vacuum carburization procedure for the X2M alloy was conducted. The original Phase I evaluations were not extensive enough to allow the vacuum carburization procedure for the X2M alloy to be fully developed and optimized. Unfortunately, the scope of the entire program did not allow for the complete optimization of the vacuum carburization procedure for X2M in Phase IV, nor for the subsequent component and full scale gear testing (Phases II and III) that was to be conducted in parallel with that done on the vacuum carburized 9310 material.



Figure 1. <u>Overview of aircraft quality high temperature vacuum</u> carburizing program.

4. EXPERIMENTAL PROCEDURES

4.1 VENDOR SELECTION

The prime vacuum carburizing vendor was the C.I. Hayes Company in Cranston, RI. C.I. Hayes has been in the forefront of vacuum heat treating technology due to its development of high quality vacuum carburizing furnaces. They are the recognized leader in the vacuum furnace industry and have been working on various high temperature vacuum carburizing procedures for many different materials. They developed a vacuum carburizing process for 9310 steel as well as the curves for the entire case thickness range. In addition, they have worked extensively with Boeing Helicopters in furthering the development of a vacuum carburizing procedure for the Vasco X2M material.

Summit Gear Company in Plymouth, MN was selected to machine all of the test gears for this program and perform various carburization heat treatments. They are a qualified source for manufacturing and heat treating aerospace gears. In addition, they have the latest C.I. Hayes Vacuum Seal Quench (VSQ) carburizing furnace available to industry, and are familiar with the 9310 and Vasco X2M alloys.

These two prime vendors, as well as the other vendors utilized in this program, are listed below with a brief description of the services provided by each:

VENDOR

SERVICES PROVIDED

C.I. Hayes, Inc. 800 Wellington Avenue Cranston, RI 02910	Development of Vacuum Carburize Cycles for Vasco X2M & 9310; Heat Treatment of Certain Test Specimens
Summit Gear Corp. 3131 Vicksburg Lane Plymouth, MN 55447	Machining of all Test Specimens and Heat Treatment of Certain Test Specimens
Teledyne Vasco P.O. Box 151 Latrobe, PA 15650	Raw Material - Vasco X2M
Carpenter Steel Company Reading, PA 19612	Raw Material - Vasco X2M and 9310
	6

Rough Machining of Billets

Stulen Machine Company 4693 Peoples Road Pittsburgh, PA 15237

Stabilization Heat Treatment of All Specimen Blanks

Litton Precision Gear Co. 4545 Western Avenue Chicago, IL 60609

4.2 MATERIAL SELECTION

The materials evaluated in this program were AISI 9310 and Vasco X2M gear steels, both of which were double vacuum melted utilizing Vacuum Induction Melting-Vacuum Arc Remelting (VIM-VAR) processing. AISI 9310 gear steel was selected because it is one of the most common steel alloys that has been used for transmissions over the past many years. Vasco X2M was selected because it is an advanced high hot hardness alloy with exceptional properties for use in higher performance/temperature transmissions. The 9310 and X2M alloys were produced according to Boeing Helicopters specifications BMS 7-249, Type III, and BMS 7-223, Type III, respectively.

X2M is presently being carburized using Boeing Helicopters patented heat treat procedure (Reference 1) and is the only high hot hardness gear steel in use today which has been fully tested and qualified for use in helicopter transmissions. Presently, X2M is being used for all critical main drive system components in the CH-47D and E Model helicopters. The CH-47D and E programs involve the modification of more than 35D aircraft. It is noteworthy that vacuum carburizing the Vasco X2M material would eliminate the need for the proprietary preoxidation process (Reference 1) which must be accomplished to uniformly endothermically carburize the steel.

4.2.1 Material and Processing - Phases I and IV

A detailed schematic showing the test requirements for Phase I is shown in Figure 2.

The nominal chemistries of the two alloys used in the program, i.e., 9310 and X2M, are shown in Table 1. The mill certifications for these alloys are shown in Appendixes A through D.

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HIGH TEMPERATURE VACUUM CARBURIZING – PHASE I



* Fifteen tests per mill heat and heat treat vendor for a total of 60 tests

Figure 2. Vacuum carburize program test requirements for Phase I.

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TABLE 1. NOMINAL CHEMICAL COMPOSITIONS OF VASCO X2M AND AISI 9310 GEAR STEELS (WEIGHT PERCENT)

Material Composition (%)

Element	<u>Vasco X2m</u>	<u>9310</u>
Carbon	0.15	0.10
Silicon	0.90	0.27
Manganese	0.30	0.55
Sulphur	0.010 Max	0.010 Max
Phosphorus	0.015 Max	0.010 Max
Tungsten	1.35	
Chromium	5.00	1.20
Vanadium	0.45	
Molybdenum	1.40	0.19
Nickel		3.25

A detailed flow diagram showing the materials and procedures used to process test specimens for Phase I is shown in Figure 3. A schematic showing the sectioning pattern for the raw material with the grain direction noted for various specimens is illustrated in Figure 4. Table 2 details the mill heats, start and finish sizes of material, as well as the work ratio, i.e., the amount of work reduction of the ingot to billet used for all the Phase I material.

The raw material was shipped from Teledyne Vasco Co. or Carpenter Steel Co. to Stulen Machine Company, where it was machined into surface contact fatigue, single tooth bending fatigue, and rotating fatigue test specimen blanks.







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Figure 4. Sectioning pattern for raw material.

MATERIAL	MILL HEAT	START SIZE	FINISH	WORK RATIO	USED TO MA MAKE* L	TERIAL ABELED	DASH NG.
Vasco X2M	5842A (Teledyne Vasco)	20 in, Round	8.25 in. Round	5.8:1	30 GR 8 STBF 20 RF	А А А	-1
Vasco X2M	86510 (Carpente Steel)	20 in. Round er	8.5 in.	4.4:1 RCS	30 GR 8 ST8F 20 RF	8 8 8	-2
9310	86670 (Carpente Steel)	20 in. Round er	8.4 in.	4.5:1 RCS	30 GR 8 STBF* 20 RF*	с с с	-3
9310	86043 (Carpente Steel)	20 in. Round 2r	8.5 in. Round	5.5:1	45 GR 8 ST8F** 20 RF**	D D D	- 4
9310	87885-2 (Carpente Steel)	20 in. Round	8.5 in.	5.5:1	8 STBF 20 RF	F F	-10 -10

TABLE 2. DETAILS OF MATERIALS USED IN PHASE I

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*GR: Gear Roller Test Specimen; STBF: Single Tooth Bending fatigue Specimen; RF: Rolling Contact Fatigue Life Specimen.

**Not tested due to poor microstructure which developed at 1,900°F vacuum carburization temperature.

Mill certifications for all the material are shown in Appendixes A through D. All specimen blanks were shipped to Litton Precision Gear Company where each was stabilization heat treated as shown in Table 3.

TABLE 3. STABILIZATION HEAT TREATMENTS

9310

Vasco X2M

1,700°F - 2 Hours	1,700°F - 2 Hours
1,850°F - 1 1/4 Hours	1,525°F - 2 1/4 Hours
011 Quench	Cil Quench
Craw 1,200 F - 3 Hours	Draw 1,000°F - 4 Hours

The primary purpose of this treatment is to produce a martensite structure in the material prior to machining to enhance machinability and to decrease distortion which could occur during subsequent carburizing and hardening.

After stabilization heat treatment, all specimen blanks were shipped to Summit Gear Company. Summit machined and ground all specimens to the SK drawing dimensions shown in Appendices E-H, and carburized and heat treated half of them. C.I. Hayes carburized and heat treated the remainder of the specimens. Several comments are given concerning the specimen machining.

- The geared roller test rolls all had 0.010-inch of grind stock per side, while the test gears had 0.005-inch of grind stock per tooth surface.
- All carburized surfaces, including the complete tooth profile, were ground to finish dimensions.
- c) All components were temper etch inspected per Boeing Helicopters BAC 5436 requirements after final grinding.
- d) After temper etching, a:1 parts were tempered for four hours minimum at a temperature 50°F below the tempering temperature of the material.
- All parts were magnetic particle inspected per Boeing Helicopters BAC 5424 requirements prior to testing.

f) Manufacturing plans were submitted to Boeing Helicopters by Summit Gear for all test components. Each plan was reviewed and approved as required.

The rough machining of test specimens from the blank pancake forgings was accomplished by hobbing all of the gear teeth. Teeth were then removed from the single tooth bending fatigue specimens as required.

Following vacuum carburizing and heat treatment, the gears were ground to final dimensions by Summit Gear, and then temper etch and magnetic particle inspected. The finished gears were then geometrically inspected and shipped to Boeing for testing.

Distortion due to heat treating was not excessive and consequently had little effect on the final grinding process. Temper etch inspection did not reveal any grinding burns on the gear teeth, indicating they were acceptable for use.

It is noteworthy that all of the processing of each component manufactured during Phase I of this program was accomplished in the same manner/procedure utilized for the production of CH-47D helicopter components. This was done such that all data obtained could be readily compared with previously obtained data and be applicable for future production transmission component processing.

4.2.2 Material and Processing - Phase II

In this Phase, spiral bevel and spur gears were produced and tested at Boeing Helicopters to evaluate the surface durability characteristics of the vacuum carburized gear tooth surfaces.

All of the gears in this phase were manufactured from the same billet materia: as those of Phase I. Test gear blanks were removed from the billets by Stulen Machine Co., after which they were processed as follows:

- 1. Forge material into individual blanks
- 2. Pough machine gear blank and stress relieve

- Final machine gear blank (bore, faces, outside diameter)
- 4. Machine gear teeth
- 5. Vacuum carburize gear teeth, Summit Gear Corp
- 6. Heat treat (harden and draw), Litton Precision Gear
- Finish grind bore and end faces, and grind gear teeth, Litton Precision Gear and Summit Gear Corp.
- 8. Bake
- 9. Perform final inspection (temper etch and geometric).

Litton final ground the spiral bevel gears, and Summit final ground the spur gears. The procedures used to test these gears is found in Experimental Procedures, Section 4. The results of testing are found in Results and Discussion, Section 5.

4.2.3 Material and Processing - Phase III

In this phase, two spiral bevel input gears, Part Number 11405245-10, serial numbers M5373 and M5372, were produced from AISI 9310 steel. Gear S/N M5373 was tested at Boeing Helicopters in a CH-47C Combining Transmission. Gear serial number 5372 was held for possible future testing and evaluation. The two gears were selected from a group of standard production spiral bevel gears at Litton Precision Gear which were being readied for production by the conventional carburization method.

The two gears were rough machined at Litton with all of the others in the group, after which they were sent to Summit Gear for vacuum carburizing. The other gears in the group remained at Litton where they were conventionally carburized. The two vacuum carburized gears were then sent back to Litton where they rejoined the group of standard production gears from which they were selected. All of the gears were then hardened and ground to final dimensions. The grinding stock remaining on the vacuum carburized gears was the same as that specified for all of the other gears which were produced by conventional carburizing. Machining, hardening, grinding, inspecting, etc., these two gears in sequence with the group of gears from which they were selected ensured that the only variables in their manufacture was the carburization process.

The vacuum carburization and hardening processes used to produce these two gears are listed below. The hardening procedure is that which is used to harden all AISI 9310 parts which are found on Boeing's Helicopters (document D210-12023-1). The vacuum carburization records can be found in Appendix I.

Carburization - Vacuum Process

- 1. Copper plate all areas not to be carburized
- 2. Load furnace at room temperature
- Carburize at 1,800°F for 65 minutes using propane-methane gas at 250 Torr to obtain an ECD of 0.035-0.055 inch
- 4. Diffuse at 1,800°F in a vacuum for 115 minutes
- 5. Quench using nitrogen gas
- 6. Subcritical anneal at 1,275°F for 150 minutes
- 7. Strip copper plate

Hardening

- 8. Nickel strike and copper plate at 0.003 inch all over
- 9. Heat to 1,150°F ±25°F for 3 hours
- 10. Heat to 1,400°F-1,500°F for 1 hour
- 11. Heat to 1,850 ± 25°F for 15-30 minutes
- 12. Quench in oil at 75-140°F
- 13. Cool to -100 to -120°F for at least 3 hours
- 14. Temper at 600°F for 2 hours
- 15. Air cool to room temperature
- 16. Temper at 600°F for 2 hours
- 17. Air cool to room temperature
- 18. Final grinding and other routine conventional processing

It should be noted that gear serial number M5372 was placed on a Rejection Report for Boeing Helicopters Materials Review Board (MRB) action due to case leakage. The leakage covered an area approximately 1.0 inch in diameter, and was located on the 2.40-inch diameter of the gear. The case leakage was the result of the copper plate which was too thin to prevent secondary carburization during hardening. Although case leakage is undesirable, it was determined that the extent and location of the leakage was not detrimental to the function of the gear, and it was released for final processing. However, to ensure that any minor or inconsequential effects of case leakage would not affect test results, the other gear, serial number M5373, was used for Phase III transmission test.

The vacuum carburized spiral bevel gear serial number M5373 was tested under Phase III of this program, and the test procedures and results are discussed in the Results and Discussion Paragraph 5.3.

4.3 DESCRIPTION OF PHASE I TEST PROCEDURES

4.3.1 Geared Roller Test

A geared roller test simulates the combined rolling and sliding conditions experienced by gears, cams, rolling element bearings, and similar machine components. It provides a means of testing materials, lubricants, and/or their interaction. The degree of sliding, the load, the lubricant temperature, and the rotating velocity are all controllable.

The heart of the geared roller test machine is a set of disks or rollers consisting of a 1-inch diameter test roll and a 5-inch diameter slave roll which are mounted on two parallel shafts and geared together in a 3.5:1 ratio. This combination of rollers and gears results in both rolling and sliding contact fatigue. The arrangement of shaft and gear is shown in Figure 5. The load is applied through a lever arrangement which is actuated by a pneumatic roto chamber.

Pressurized air from an external source controls the load to within ± 3 percent. The actual load is determined by a calibrated strain gage while a pressure gage is used to check the stability of the load during periods of extended operation. The load is converted to a Hertzian stress by the relationships found in Reference 2.

The following parameters were employed during all testing:

- a) Load 450,000 psi Hertz compressive stress
- b) Lubricant MIL-L-23699 of1
- 17



ITEM	PARTNAME	PART NUMBER*
1	LOWER SHAFT - GEAR END	201-C-086
2	UPPER SHAFT GEAR	201-C-114
3	UPPER SHAFT	201-C-144
4	TEST SPECIMEN (1-INCH D:AMETER)	201-8-047
5	LOWER SHAFT	201-C-058
6	SLAVE ROLL (5-INCH DIAMETER)	201-B-229

* ALL PART NUMBERS ARE THOSE LISTED IN THE TEST MACHINE BROCHURE FROM THE MANUFACTURER GEARS 1 AND 2 WHICH HAVE 16 AND 56 TEETH, RESPECTIVELY, PROVIDE A 3 5011 RATIO

Figure 5. Geared roller test setup
- c) Rotating velocity 900-1000 rpm
- d) Test oil temperature 200 ± 10°F.

To insure proper "break-in" of the lubricating system, a dummy specimen was run for a least 148 hours prior to the start of the first test. To renew the MIL-L-23699 oil during the test, one-half of a gallon of oil was changed for each 250 hours of total test time. A runout of any one specimen was considered to be the accumulation of 10×10^6 cycles without failure. None of the runout specimens were retested. Testing was accomplished by randomizing both the specimen tested and the test machine utilized. A total of three machines were employed for testing, serial numbers 3A437, 3A438, and 3A439.

Testing proceeded on a 24-hour basis. A failure was detected by a sensitivity switch which monitored the level of vibration of the test machine. When a spalling and/or pitting failure occurred, the vibration in the machine increased which resulted in machine shutdown. A time counter (hours) was connected to this sensitivity switch and became disconnected at machine shutdown.

The data recorded for the geared roller tests consisted of:

- part number
- serial number
- applied load
- test duration

A summary of these tests is discussed in paragraph 1.3.

4.3.2 Single Tooth Bending Fatigue Test

Before any new development can be incorporated into a helicopter gear box, it must be thoroughly evaluated from two viewpoints. Obviously, the first consideration is determining whether or not the specific advantage claimed is actually achieved. The second, and no less important, consideration is a determination of the side effects of the proposed new development. In the case of vacuum carburizing, one of these potential ϵ fects is the structural integrity of the gear teeth. Since the strength of gear teeth, in general, is

a subject of great concern in the design of helicopter gear boxes, considerable testing of the strength of various gear designs and materials has been accomplished at Boeing Helicopters. Standardized test setups and methods have been developed and are routinely used for evaluating the strength of gear teeth. This standardized evaluation technique was applied to full sized test gears of two different materials (9310 and Vasco X2M) with tooth proportions typical of a final drive planetary system. Each gear was carburized by the vacuum process. The purpose of this testing was neither to confirm nor identify some advantage of the process. Rather, the goal was to insure that the cost, time, and consistency improvements obtained through the vacuum carburization process were not obtained at the expense of part strength.

With this background in mind, it is clear that a satisfactory test result is simply the identification of no statistical difference in the strength of identical parts carburized by either conventional or vacuum processes. Any gain which may be obtained in strength is purely a beneficial side effect.

4.3.2.1 Single Tooth Bending Fatigue Specimen Design - The design of the test gears (see Appendix G) utilized in this program is within the experience range of helicopter main transmission power gears in pitch diameters, diametral pitch, pressure angle, and profile modifications. All tolerances and records for the manufactured test specimens conformed to the appropriate Boeing Helicopters production specifications. Each dash number grouping of test gears (for instance SK29572-1 and SK29572-2 are two different dash number groups of the same basic part number) was heat treated in a single batch to minimize variations within batches.

The test gears were manufactured with a total of 32 teeth. However, for testing purposes, every other set of four teeth was removed to permit placement of the test specimen within the test fixture arrangement, thus allowing four gear teeth on each gear to be subjected to fatigue testing. The gear teeth selected for testing were spaced in such a manner as to eliminate any possible effects of previously incurred fatigue failures on adjacent test teeth. The processes used to manufacture these gears are discussed in Paragraph 4.2.1. 4.3.2.2 Test Apparatus - The gear specimens were tested on a nonrotating single tooth bending fatigue test fixture (Figure 6) designed by Boeing Helicopters for use on a Baldwin-Lima Hamilton IV-20 Universal Fatigue Machine. The test machine is capable of developing total loads up to 16,000 pounds (8,000 pounds steady and 8,000 pounds alternating load) at a frequency of 1,200 cycles per minute.

The test fixture was specifically designed to conduct nonrotating bending fatigue testing, and permits application of a cyclic load to one gear tooth at a time. Present analytical methods established by the American Gear Manufacturers Association (AGMA) rate the maximum bending strength of a gear at the critical section when loaded at the Highest Point of Single Tooth Contact (HPSTC).

The design of the fixture is such that the test gear tooth is loaded at the highest point of single tooth contact (HPSTC), which is based on a one-to-one gear ratio. The location of the load anvil at the HPSTC is maintained during setup for each gear tooth by maintaining a constant static height (with a load of approximately 100 pounds) on the load anvil, thus assuring loading through the load angle at the HPSTC due to the geometry of the test fixture.

Load on the test gear tooth was transferred from the test machine to the gear tooth through a load link. All load links were instrumented with strain gages and connected to an oscilloscope to permit monitoring of the gear tooth load during testing.

4.3.2.3 Testing Technique - The test specimen was mounted in the gear fatigue fixture in the manner shown in Figure 6. The height of the load anvil was adjusted to the required position, as determined by gear and fixture geometry, and the reaction anvil was then moved into position on the reaction tooth. Load anvil height was rechecked with a compressive load of approximately 100 pounds. The specimen was then ready for test load application if the load anvil height was found to be within the correct tolerance.

The steady load was maintained approximately 100 pounds above the alternating load during all test runs so that impact loading of the test tooth was



Figure 6. Single-tooth bending fatique test fixture.

avoided. The load was unidirectional in all cases as would be typical of a simple gear mesh. The 100 pounds preload maintained on the gear teeth during testing represents less than 2 percent of the total load on the gears and was therefore considered a zero load.

Since the test gear teeth were loaded at a rate of approximately 1,200 cycles per minute, no localized heat buildup was noticeable in the fillet area of the test gear tooth due to the constant cycling of stress. Heating of the fillet did not occur throughout the entire loading range, no matter now long or how little a particular gear tooth was cycled before failure. The constant temperature maintained by the test gear specimens during testing precluded the effect of gear tooth root fillet temperature increase on the test results.

A small amount of moly grease was applied to the load and reaction anvils at the tooth contact points but no other lubrication was provided.

To insure accurate data, one tooth in each group of test gears was instrumented with a strain gage located at the critical section in the tooth fillet region. A calibration curve was then developed so that tooth bending stress could be measured directly.

Each specimen was run continuously until failure or runout (6 x 10° cycles), whichever occurred first. Failure is defined as a crack length of approximately 0 25-inch. Testing was terminated either manually by the test technician upon observance of a crack or occurrence of a runout, or automatically (during unattended running) by limit switches. The 0.25 inch crack length was chosen as a failure criteria to be consistent with the previously acquired data with which the current data was compared.

4.3.2.4 Gear Stress Calculations - The gear stresses presented in this report were calculated by a computer program based on AGMA standards for rating the strength of spur gear teeth. Calculation of the geometry factor for the test gear was based on an assumed gear ratio of one-to-one.

Based on the AGMA standards, the equation for calculating the bending stress at the critical section of a gear tooth when loaded at the highest point of single tooth contact is:

AGMA methods include factors to account for dynamic loading, misalignment, etc. In these analyses, all of these factors were taken as unit. By utilizing the engineering drawing (Appendix G) data for the test gears, equation 1 can be reduced to a function of tangential tooth load as follows:

 $S_{T} = 27.14 W_{L}$ (2)

The above stress calculations are provided for reference only, since actual tooth bendiny stresses were measured during the test program using strain gages. Additional information concerning the statistical method for analyzing this date is shown in Appendix J.

4.3.2.5 Test Data - The data recorded for the single tooth bending fatigue tests consisted of:

- Part Number

- Serial Number

- Test Tooth Number
- Applied Load, Steady and Alternating
- Cycles to Failure (or Runout)
- Crack Length
- Failure Mode

A summary of the single tooth bending fatigue data is discussed in Paragraph 5.1.4.

4.3.3 Scoring Test

As was the case with the single tooth bending fatigue strength, the scoring capacity of vacuum carburized gears must also be evaluated to insure that the process did not, somehow, produce an unknown side effect which reduces the ability of the gears to resist failure by scoring.

Scoring is a very significant problem in the design of helicopter gear systems. Under conditions of high speed and heavy load, the thin oil film which normaily separates the mating gear tooth surfaces is sometimes destroyed. When this happens, the asperities of the tooth surfaces come into contact and generate enough heat to allow them to instantaneously weld together on a microscopic scale. Continued rotation of the gear causes these micro welds to be pulled apart and the resulting sliding motion along the tooth flanks produces the scratches which typify a scoring failure. Scoring is not a fatigue phenomena. If it is to occur at all, it will occur in a very short time (usually 10 or 20 minutes) of operation. If it does not occur within 10 or 20 minutes it will never occur as long as the operating conditions remain constant. Several theories have been proposed to explain this phenomena for aerospace gears which are operating with synthetic oils. The critical temperature theory, first proposed by Blok (Reference 3), shows the greatest correlation with actual test results. This theory states that the instantaneous temperature of the contact point at any time is a function of the material properties of the gears and the oil, as well as the combination of sliding and contact pressure which exists at that point on the tooth surface. When the instantaneous contact temperature due to these combined effects reaches a critical or "flash" temperature, the film of oil is

destroyed and scoring occurs. The parameter used to evaluate the scoring behavior is thus known as the flash temperature.

The relative scoring behavior of various materials and lubricants is a key factor in the design of helicopter transmissions. Because this behavior is a key factor, considerable score testing has been accomplished at Boeing Helicopters, where standardized test machines and methods have been developed and are routinely used for such test programs.

This standardized evaluation technique was applied to full sized gears in this program which were carburized by the vacuum process. The tooth proportions were typical of a final drive planetary system. The purpose of this testing was neither to confirm nor identify some advantage due to the process. Rather, the goal was to insure that the cost, time, and consistency improvements obtained through the vacuum process were not obtained at the expense of the scoring capacity of the parts.

With this background in mind, it is clear that a satisfactory test result is simply the identification of no difference in the scoring behavior of identical parts carburized by either conventional or vacuum processes. Any gain which may be obtained in this area is purely a beneficial side effect.

4.3.3.1 Scoring Test Specimen Design - As with the single tooth bending fatigue test specimens, the design of the scoring test gears (Appendix H) utilized in this program is within the experience range of helicopter main transmission power gears in pitch diameter, diametrical pitch, pressure angle, and profile modifications. All tolerances and records for the manufactured test specimens conformed to the appropriate Boeing Helicopters production specifications. Each dash number grouping of test gears (for instance, SK29571-1 and SK29571-2 are two different dash number groups of the same basic part number) was heat treated in a single batch to minimize variations within batches.

These test gears were designed to simulate a typical first stage planetary system sun-planet mesh for the final drive of a medium to large helicopter. By way of comparison, they are quite representative of that set on either the CH-46 or CH-47 helicopter. The processes used to manufacture these gears are defined in Paragraph 4.2.1.

4.3.3.2 Test Apparatus - The score testing was conducted in the Boeing Helicopters Gear Research Test Facility, Figure 7, which is located in the Transmission Assembly and Test Building. This facility is designed to test full sized, representative, test specimens. It will accommodate spur, helical, and spiral bevel gears. There are two identical test rigs in the facility, each of which may be easily configured for a wide variety of test programs. The standard scoring test configuration (6 inch center distance, 1/2 inch face width, overhung mounted gears with isolated oil supply) was used for this program, Figure 8. This setup incorporates provisions for controlling center distance, speed, oil temperature, torque, and oil flow. The system is a regenerative (four-square) design using one gearbox as the slave unit and one gearbox as the test unit. Gear mountings ware designed to be rigid and stable under all loading conditions, with through-bored housings for maximum accuracy.

To facilitate short-term operation for scoring tests, the test stand design provides for testing outboard of the main gear housing and allows for rapid assembly and removal of the test specimens and good accessibility for frequent visual inspection of the test gears, as Figure 9 shows. This test stand arrangement has a separate lubricating system for the test housing with heating and cooling capabilities and direct oil flow measurement. Lubrication is directed to the test gears by individual, externally cooled oil jets which can be directed on the in-mesh side, the out-of-mesh side, or both sides simultaneously. This configuration also permits control of the oil flow rate, oil inlet temperature and operating torque while maintaining a constant speed. Power is supplied by an electric motor driving the input shaft through a toothed belt arrangement.

All test parameters as well as the general test stand operation are monitored continuously from the test stand control center, which is located just outside of the test cell.





Figure 8. <u>Scoring test setup</u>.



Figure 9. <u>Research test stand overhung configuration.</u>

Since scoring is sensitive to the oil temperature as well as oil type and gear material, a sophisticated temperature control system is incorporated in the test setup.

A single jet on the out-of-mesh side of the gear set is utilized in (Figure 9) to supply 0.32 GPM of oil at 40 psi to the test gears. The bearings supporting the test gears are sealed from this oil flow and are separately lubricated. By maintaining the test oil (MIL-L-23699) in a temperature controlled, heated tank (Figure 10), the temperature to the test gear jet is controlled to 200°F. Very fine control over temperature is maintained by an electronically controlled in-line heater located just before the test gear jet. The test oil and both of the slave box oils are cooled and filtered (Figure 11) after leaving the boxes. A 12 micron filter element was used.

4.3.3.3 Testing Technique

The primary scoring test variables were shaft torque and oil inlet temperature. Gear tooth load, a function of shaft torque, was applied through a lever system at the beginning of each test run. Torque levels were observed on a Strainsert SR2 instrument at the seginning and conclusion of each test run.

Deviation from the initial target torque was controlled within ±5 percent at test startup. The torquemeter was calibrated, through a load spectrum of 0 to 40,000 inch-pounds before and at the conclusion of the test program. Recalibration curves agreed with the initial curve within 2 percent. Test time (cycles) was determined by a log record of running time and an elapsed-time meter in the test stand console. Power was supplied by a 100 horsepower electrical motor driving the input shaft through a toothed belt arrangement, which maintained the input pinion speed at 3,660 revolutions per minute. Test runs were initiated only after stabilization had been achieved.

Although the theory that a gear set's load capacity may be improved by incremental loading techniques has been advanced from time to time, it should be noted that it only holds true for relatively soft gears and/or those with very rough initial surface finishes. Carburized and ground gears such as



Figure 10. Insulated, temperature controlled test-oil supply tank.



Figure 11. Cooling, lubrication, and filtering system.

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these may also benefit from such incremental loading if a special high load capacity oil is utilized. None of these were the case for these gears, thus, run-in yields insignificant advantage. With this in mind, each test gear set was step loaded as shown in Figure 12 until a scoring failure was obtained. A scoring failure was obtained in every case. There were no runouts and no other types of failures occurred.

4.3.3.4 Gear Stress and Flash Temperature Calculations – The gear stresses and flash temperatures presented in this report were calculated by a computer program based on AGMA standards for rating the strength, durability, and scoring hazard of spur gear teeth. Equation 1 (Paragraph 4.3.2.5) is the basic bending stress equation. By utilizing the information shown on the engineering drawing (Appendix H), this basic equation can be reduced to a function of tangential tooth load or shaft torque for these score test gears, as follows:

$$S_t = 24.17 \quad W_t = 8.06 T$$
 (3)

- where: S_t = Calculated tensile stress at critical section, PSI
 - Wt = Transmitted tangential load, pounds
 - T = Shaft torque, in-1b

While AGMA methods include factors to account for dynamic loading, misalignment, etc., in our analysis, all of these factors have been taken as unity. The test system configuration and gear quality are such that these effects are negligible.

The basic contact stress equation is:

$$S_{c} = C_{p} (WT/FdI)^{0.5}$$
(4)

where: S_c = Calculated contact stress, psi



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Figure 12 Scoring test procedure.

Cp = Material factor (2300 for steel spur gears)
WT = Tangential load, lb
F = Net face width, inch
d = Pinion pitch diameter, inch
i = Geometry factor

As was the case for the bending stress equation, equation four can also be reduced to a function of tangential tooth load or shaft torque only, for these test gears.

$$S_{c} = 4628 W_{T}^{0.5} = 2672 T^{0.5}$$
 (5)

Finally, the parameter of greatest interest in this test, the flash temperature, is calculated by:

$$T_F = T_I + (W_T / F)^{0.75} (50/(50-S)(Z_t n_p^{-0.5})/(P_d 0.25)$$
(6)

where: T_f = Flash temperature scoring index (^OF)

- T₁ = Initial temperature (^OF), (oil jet temperature for these test gears is virtually the same as the gear blank initial temperature)
- S = Surface finish (RMS)
- Zt = Scoring geometry factor
- n_p = Pinion speed (RPM)

As with the single tooth bending fatigue calculations, by utilizing the specific geometry of these test gears, this equation can be reduced to a function of shaft torque or tangential tooth load.

$$T_{F} = 200 + 0.735 (W_{T})^{0.75} = 200 + 0.322 (T) 0.75 (7)$$

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The critical point for scoring occurs just below the lowest point of single tooth contact, thus the first signs of scoring should occur at the tips und/or flanks of the teetn.

4.3.3.5 Test Data - The data recorded for the score tests consisted of:

- Part Number
- Serial Nulleer
- Applied Shaft Torque
- Loaded Side Designation
- Inlet Test Oil Temperature
- Gear Tooth Condition
- Run Time
- 011 Flow Rate
- 011 Pressure
- Slave Gearbox Data (Pressure, Temperature, etc).

A summary of the score test results is discussed in paragraph 5.1.5.

4.4 DESCRIPTION OF PHASE II TEST PROCEDURES

The design of the spur and spiral bevel test gears used in Phase II of this program was within the experience range of helicopter main transmission power gears in pitch diameter, diametrical pitch, pressure angle, and profile modification. Processing procedures, tolerance parameters, and recording requirements conformed to the appropriate Boeing Helicopters production specifications.

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With these design criteria in mind, and to determine the surface load capacity of the heat treatment process under investigation, the test gears for this program were designed with the following specific parameters.

4.4.1 Spur Gears

The gear r io (Mg) of 1.67 to 1.00 was selected as the most practical for a 6.00-inch center distance while maintaining a reasonable volt circle for mounting purposes. The roll angle to the first point of contact on the pinion member was maintained below 7 degrees. The pinion member was designed with a short addendum (0.06-inch) and the gear member with a long addendum (0.22-inch). The resulting profile contact ratio was 1.13 minimum, which is below normal design practice. Pinion input speed was selected as 910 revolutions per minute.

Kinetic analysis of the design parameters, using an existing Boeing Helicopters computer program, indicated a very high specific sliding (slide/roll ratio) value at the first point of contact on the pinion member. The specific sliding value at this point was considerably higher than the value for any other point along the tooth profile, indicating a high probability of experiencing surface type failures in the pinion dedendum. The general design parameters are listed in Table 4. The actual dimensions of the spur gears are shown in Appendix K.

TABLE 4. TEST SPECIMEN GENERAL DESIGN PARAMETERS - SPUR GEARS

Member	Material (Steel)	Pitch Diameter (inches)	Diametrical Pitch	Gear Ratio	Pressure Angle (degrees)	Face Width (inches)	
Gear	AISI 9310	7.500	5.333	1.67	20	0.500	
Pinion	AISI 9310	4.500	5.333	1.67	20	0.500	

The final design parameters selected for the gear test specimens were specifically chosen to increase the pitting probability; consequently, they are not representative of typical aircraft design practice.

4.4.2 Spiral Bevel Gears

The spiral bevel gear test rig, unlike the spur rig is designed to simulate the CH-47C engine transmission bevel gear set. The rig will accommodate two bevel gear configurations. The first is an actual production set of CH-47C gears. The second is a less expensive set of slightly smaller test gears which simulate but do not actually duplicate the CH-47 engine box gears.

Since these gears simulate an actual aircraft application, it was not possible to bias the design to produce only pitting failures.

The final design parameters for the spiral bevel gears are shown in Table 5. The actual dimensions are shown in Appendix L.

TABLE 5. TEST SPECIMEN GENERAL DESIGN PARAMETERS - SPIRAL BEVEL GEARS

Memt er	Material (Steel)	Diameter (inches)	Diametric Pitch	al Gear Ratio	Pressure Angle (degrees)(Spiral Angle degrees)	Face width (inches)
Gear	AISI 9310	7.372	5.833	1.72	22.5	26	1.43
Pinion	AIST 9310	6.000	5.933	1.72	22.5	26	1 43

4.4.3 Spur and Sprial Bevel Gear Testing

4.4.3.1 Test Apparatus - The gear specimens were tested on a Boeing Helicopters regenerative (four square) load test stand. These test stands were specifically designed and constructed to conduct rotating load test programs for gear research and development. The spur test machine is capable of operation with three center distance options and provisions for control of torque, oil temperature, and quantity of oil. Lubrication of all gear meshes and bearings is provided by individual oil jets. To facilitate short term operation and surface durability type testing, the design of this test stand includes the provision for testing outboard of the main gear housing, as shown in Figure 13. This feature provides for rapid assembly and disassembly of the test specimens, with improved accessibility for frequent visual inspection. This test stand configuration has a separate lubrication system with heating and cooling capabilities and direct oil flow measurement. Lubrication is directed to the test gears by individual externally cooled oil jets, which can be directed on the in-mesh side, out-of-mesh side, or both sides simultaneously. This configuration also permits control of oil flow rate, oil inlet temperature, and operating torque, while maintaining a constant speed.

The spiral bevel test rig is almost identical to the spur rig in design and operation. In fact, either stands may be used for testing spur helical or bevel gears, depending on which of the interchangeable test heads is mounted on the rig.

4.4.3.2 Testing Technique - The primary test variables were shaft torque and oil inlet temperature. Gear tooth load was a function of shaft torque, which was applied through a lever system at the beginning of each test run. Torque levels were observed on an SR2 Strainsert instrument at specified intervals, and recorded. A final torque reading was taken and recorded at the conclusion of each test run. Deviation from the initial target torque was controlled to plus or minus five percent at start-up and within the initial two percent during the individual test runs.

The torquemeter was calibrated prior to and at the completion of the test program. Recalibration curves agreed within two percent with the initial calibration. Test time (cycles) was determined by a log record of running time and an elapsed time meter located in the test stand console. Power was supplied by an electric motor driving the input shaft through a toothed belt arrangement, maintaining the input pinion speed at 910 Revolutions Per Minute for the spur gears and at 3,660 revolutions per minute for the bevel gears.



Figure 13. Test stand for surface durability testing.

Input oil to perature for the test gearbox was maintained at less than 135°F with input oil pressure of 55 ±5 per square inch. The oil used for lubricating the test gearbox was MLL-L-23699. Testing technique for this test program consisted of rotating load tests at each of the specified load levels for a maximum of three million cycles (or failure). Successful completion of a particular test run for three million cycles was considered as a test runout data point. This runout was then considered to be below the fatigue endurance limit. Prior to conducting the test runs, the lubricating oil was circulated until the oil-temperature stabilized. Jet lubrication was provided on the out-of-mesh side for the test gear mesh.

The test procedure for all test gears in this program was the same, and consisted of the following sequence:

- Conduct static pattern checks at the 50 percent and 100 percent load levels for load distribution evaluation.
- 2. Complete the test load schedule by conducting rotating tests for a maximum of 3×10^6 cycles (or failure) at each of the specified load levels.

4.4.3.3 Gear Stress Calculations - The gear stress levels presented in this report were calculated by an existing Boeing Helicopters computer program which uses AGMA (American Gear Manufacturers Association) standards in the analysis.

AGMA rates the bending strength of spur gears as follows:

$$St \approx \frac{W_t \times K_o \times P_d \times K_s \times K_m}{K_v \times F \times J}$$
(8)

Where

Wt = transmitted tangential load (pounds)
Ko = overload factor

Kv = dynamic factor Pd = diametrical pitch F = face width Ks = size factor Km = load distribution factor J = geometry factor

For the spur test gears utilized in this program, the following was assumed:

 $K_0 = K_V = K_s = K_m = 1.0$

For the bevel test gears, the following was assumed:

 $K_0 = K_V = 1.0$ $K_5 = 0.64$ $K_m = 1.1$

Then,

AGMA rates the surface durability of spur gears as follows:

$$SC = CP\left(\frac{W_{t} \times C_{o} \times C_{s} \times C_{m} \times C_{f}}{Cv \times d \times F \times I}\right)^{0.5}$$
(11)

where

Sc = calculated contact stress number at the lowest point
 of single tooth contact
Cp = elastic coefficient (2300 for spurs, 2800 for bevels)
Wt = transmitted tangential load at operating pitch
 diameter (pounds)
Co = overload factor

Cv = dynamic factor
d = pinion operating pitch diameter (inches)
F = face width (inches)
Cs = size factor

- Cm = load distribution factor
- I = geometry factor
- Cf = surface condition factor

For the spur test gears used in this program, the following was assumed:

Co = Cv = Cs = Cm = Cf = 1.0

For the bevel gears:

Co = Cv = Cs = Cf = 1.0 Cm = 1.1

then,

$$S_c = 6520 (Wt)^{0.5}$$
 (spur gears) (12)
 $S_c = 3644 (Wt)^{0.5}$ (bevel gears) (13)

4.4.3.4 Test Data - The basic 100 percent spur gear test load level for this program was established as 3094 inch-pounds of pinion torque resulting in a pinion bending stress of 44,688 psi and a contact stress at the lowest point of single tooth contact of 241,768 psi.

The basic 100 percent bevel gear test load level for this program was established as 12,606 inch-pounds of pinion torque resulting in a pinion bending stress of 36,948 psi and a contact stress of 236,277 psi.

The criteria for definition of a failure for all of the test gears used in this experimental program were established as follows:

A. A minimum of one pit per tooth, on each of three nonadjacent teeth having a minimum dimension of 1/16 inch shall constitute a failure.

- B. The appearance of a crack anywhere on the test parts. The results of this testing are discussed in paragraph 6.
- 4.5 DESCRIPTION OF PHASE III TEST PROCEDURES

The test gear design and fabrication and all test procedures are described under paragraph 5.3.

5. RESULTS AND DISCUSSION

5.1 PHASE I - VACUUM CARBURIZE DEVELOPMENT

5.1.1 Development of the Vacuum Carburize Cycle for the Vasco X2M Gear Material

To develop the vacuum carburize cycle for the Vasco X2M material, approximately 50 standard heat treat test slugs were obtained from Litton Precision Gear and sent to C.I. Hayes. Litton had prepared these slugs by machining a gear tooth profile in a cylindrical bar. The profile was representative of gear teeth presently machined by Litton for CH-47D helicopter transmission components. Use of this standard type of specimen allowed an accurate assessment of the amount of carburization throughout the tooth profile to be made.

Vasco X2M is conventionally carburized at 1,700°F. To make full use of the inherent high process temperature and enhanced diffusion advantages of the vacuum carburization process, the initial temperature evaluated was 1,750°F. This temperature is 100 degrees below the hardening temperature and below the grain coarsening temperature of the alloy. The remainder of the heat treat procedure, i.e., the hardening cycle, was the same as the conventional cycle for this material. The conventional hardening cycle was used so that the carburizing method and temperature would be the only variables to observe when comparing conventional data previously generated to the data obtained in this program.

In addition, use of the standard hardening cycle would allow a much easier implementation of the entire process into industry.

To develop the vacuum carburization procedure for X2M, C.I. Hayes personne? reviewed previous Boeing Helicopters IR&D data obtained from vacuum carburizing the Vasco X2M material. Then, various cycles were evaluated during the developmental stage of this program, as listed in Table 6. Evaluation of the data from these runs led to the formulation of the cycle listed below. At least three heat-treat test slugs were used per run (a run being one complete

Parameter	1	2	3	4	5
Pump Down		10 min	10 min	••	
Heat to 1,900°F	28 min	50 min	50 min		
Soak @ 1,900°F	35 m1n	35 min	20 min	35 min	35 min
Cool to 1,800°F	1,750°F for 22 min	1,750°F for 10 min	6 min		
Soak @ 1,800°F	1,750°F for 10 min	1,750°F for 10 min	10 min	10 min	10 min
Carb @ 1,000°F	$1,750^{\circ}F$ $C_{3}H_{8}$, 150 Torr at 40 ft ³ /hr	1,750°F C ₃ H ₈ ,125 Torr at 40ft ³ /hr	60 min CH ₄ ,255 at 75 ft ³ /hr	150 min CH ₄ ,250 Torr at 75 SCF/hr	150 min CH ₄ ,250 Torr at 75 SCF/hr
Diffuse @ 1,800°F	1,750°F for 4 min	1.750°F for 4 min	5 m1n	255 min	315 min
Gas Quench	64 min	60 min	60 min	Yes	Yes
Temper 0 1,250°F	No 90) min, N ₂ 90	0 m1n, N ₂	90 min	90 min
Heat to 1,850°F	31 mtn	45 min	45 min	~-	
So ak @ 1,85 0°F	20 min	35 min	35 min	30 min	30 m1n
011 Quench	35 min	35 min	35 min	Yes	Yes
Freeze 0 -120°F			Yes	Yes	Yes
Temper @ 600°F 2+	2		Yes	Yes	Yes
ECD'S	0.040 in (Hayes 1.1% C-Hayes 0.045050" 1.1% C-BH	0.045 in. Hayes O per BH	0.047 in. Hayes	0.080 in. Hayes	0.085 in.
Microstructure	Carbide Network Broken	Heavy Carbides	Heavy Carbides- Hayes	Carbides Ø 0.5 Hayes	Broken Network Hayes

TABLE 6. VACUUM CARBURIZE - X2M EXPERIMENTAL PROCEDURES

vacuum carburize and harden cycle), and ten to twelve runs were made using this cycle.

- a) Place test slugs in VSQ vacuum carburize furnace.
- b) Heat to 1,900°F.
- c) Soak @ 1,900°F for 35 minutes.
- d) Cool to 1,750°F.
- e) Carburize @ 1750°F for 90 minu es, Suit -250 Torr,
 75 SCF/NA; Target effective case depth (ECD) was
 0.040-0.060 inch.
- f) Gas quench.
- g) Reheat to 1,850°F for hardening.
- h) Hold @ 1,850°F for 20 minutes.
- i) 011 quench.

Each run was made to simulate an actual production cycle by placing several hundred pounds of scrap in the furnace with the test samples. After completion of a run, the samples were analyzed by both Hayes and Boeing Helicopters laboratory personnel.

The metallurgical characteristics shown on Table 7 of the material processed by the cycle above were assessed at the flank, root, and root fillet positions of the test slugs. All of these characteristics were considered acceptable.

TABLE 7. METALLURGICAL TEST RESULTS OF VASCO X2M VACUUM CARBURIZED AT 1,750°F

Property	Flank Root		Root Fillet	
Carburization Uniformity	Acceptable	Acceptable	Acceptable	
Effective Case Depth	0.048-in.	0.044-1n.	0.044-in.	
Surface Hardness*	R/C 63	R/C 62	3/C 62	
Core Hardness	R/C 42	R/C 42	R/C 42	
Surface Carbon Content**	1.2%	1.1%	1.0%	

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* Direct Rockwell C Scale.

**Determined by Electron Beam Microprobe Analysis.

The hardness versus depth curve obtained from the same test slug is given in Figure 14.

A typical percent carbon versus depth curve, obtained by electron beam microprobe analysis on a cross section through the test slug with a deep ECD of 0.070-0.110 inch, is shown in Figure 15.

Even though the metallurgical data indicated that acceptable results were obtained from this vacuum carburization run, the following changes were made to the process to crient it to production processing:

- An increase in the Vasco X2M steel carburizing temperature from 1,750°F to 1,800°F.
- b) A longer diffusion time during carburizing was added.
- c) A stress relief anneal cycle at 1,250°F was added.

The carburizing temperature was increased to improve furnace cycle times, while the diffusion time was increased to enhance the carburized case microstructure by reducing the amount of carbide at the surface of the part. (Note that the carbon content at the surface, Figure 15, is approximately 1.5 percent. This is higher than the desired 0.9 to 1.2 percent optimum carbon content). The purpose of the stress relief anneal was to facilitate standard CH-47 manufacturing techniques where it sometimes becomes necessary to machine gears between the carburizing and hardening cycles. This anneal essentially lowers the hardness of the carburized surface to Rockwell C (R/C) 28-35, which allows machining of the carburized areas, if required.

Incorporation of these changes resulted in the vacuum carburize cycle detailed below. Note that use of the vacuum carburizing procedure negates the use of the preoxidation process, which is required to overcome differences between surface and bulk diffusion rates when carburizing by conventional methods. This cycle contains the standard hardening sequence used for X2M.









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- e) Place test slugs, test part, etc., in VSQ vacuum carburize furnace.
- Evacuate chamber to 500 Mu vacuum level.
 (The heaters in the VSQ furnace do not come on until a vacuum of 500 Mu is reached.)
- c) Heat to 1,900°F.
- d) Soak @ 1,900°F for 35 minutes.
- e) Cool to 1,800°F.
- f) Scak at 1,800°F for 10 minutes.
- *g) Carburize 150 minutes at 1,800°F CH₄ 250 Torr -75 SCF/HR; Target ECD of 0.070-0.110 inch.
- **h) Diffuse for 315 minutes at 1,800°F.
 - 1) Gas guench.
 - j) Stress relieve at 1,250°F for 90 minutes in vacuum.
 - k) Reheat to 1,850°F for hardening (500 Mu vacuum).
 - 1) Hold @ 1,850°F for 30 minutes.
 - m) Oil quench.
 - n) Deep freeze at -120°F for 120 minutes at temperature.
- o) Double temper at 600°F for 2 hours each.

*Carburizing time for STBF & scoring tests is 60 minutes. **Diffusion time for STBF & scoring tests is 15 minutes.

The above cycle was utilized to vacuum carburize all of the X2M geared roller test specimens, Part Numbers SK20895-1 and -2. The single tooth bending fatigue and scoring test specimens were processed by the same cycle, with the exception of those items denoted by the asterisks.

Both C.I. Hayes and Summit Gear Co. processed these gears according to this process. Evaluation of test slugs processed with the geared roller specimens revealed the metallurgical characteristics shown on table 8.

TABLE 8. METALLURGICAL TEST RESULTS OF VASCO X2M VACUUM CARBURIZED

Property	<u>Result</u>
Uniformity	Acceptable
Effective Case Depth	0.086-in
Surface Hardness	R/C 63.5
Surface Carbon Content	N/A

The case and core microstructures of the test slugs are shown in Figures 16 and 17. The case microstructure consisted of a relatively heavy, interconnected carbide network interspersed in a tempered martensite matrix while the core microstructure consisted primarily of tempered martensite with less than two percent free ferrite. The heavy carbide network observed at the surface of the test slugs was considered rejectable per Boeing's conventional carburizing specification D210-10342-1 (Reference 1). However, since the test slug had not been ground and the geared roller test samples were to be ground, it was considered that the heavy network at the surface would be removed during grinding. As a result, the test components were heat treated.

It is important to note here that this process was successful in producing the desired metallurgical case depth characteristics. In addition, the use of the proprietary preoxidation process (Reference 1) was not required nor was it utilized. Also, a time savings of approximately 60 percent was obtained by vacuum carburizing (approximately 10 hours) compared to conventional carburizing (approximately 24 to 30 hours). This time savings can also be viewed on the basis that the total time for vacuum carburization and hardening was approximately 21 hours compared to the 24 to 30 hours required solely for conventional carburization.



100X



400X

Figure 16. Case microstructure of vacuum carburized X2M consisting of heavy, interconnected carbides in a tempered martensite matrix. The asground condition is shown.


Figure 17. Core microstructure of vacuum carburized X2M consisting of tempered martensite. Arrows indicated areas of the free ferrite

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As a result of the development of the vacuum carburize procedure for X2M, C.I. Hayes completed the formulation of the empirical equations to control furnace time in relation to effective case depth. The equations take into account the possibility of carburizing, diffusion, and hardening at different temperatures and times. The report written by C.I. Hayes covering this work is shown in Appendix M.

Following the above evaluations, gear roller test specimens were fabricated and tested. Initial test results revealed that the microstructural constituents, i.e. primarily carbide networks, had not been ground off of the specimens as expected. As a result, the gear test 'ata was considered invalia and not representative of aircraft quality processin. These test results are discussed in paragraph 5.1.3. Due to the scope and schedule of the program, further efforts to develop an optimized vacuum carburization procedure for X2M were continued in an originally unplanned Phase IV, which was developed, submitted to AMTL, and approved. The results of this Phase IV work are discussed in paragraph 5.4.

All of the 9310 vacuum carburization development work was completed in Phase I, as detailed in the next section.

5.1.2 Development of the Vacuum Carburize Cycle for the 9310 Gear Material

As stated previously in this report, the procedure for carburizing 9310 steel in a vacuum had been developed previously by the C.I. Hayes Company. Effective case depth (R/C 50 depth) versus time in the furnace for both carburizing and diffusion cycles had also been developed. Hayes' work encompassed vacuum carburizing temperatures of 1,650°F to 1,900°F, while varying surface carbon present from 0.6 to 1.10. This data is given in Appendix N.

In accordance with the program requirements, the vacuum carbinizing temperature first evaluated was 1,900°F. (It was later found that 1,800°F was optimum, as discussed below.) The 1,900°F temperature was 200F above the conventional carbunizing temperature of 1,700°F. As with the Vasco X2M alloy, the subsequent hardening heat treat procedure for 9310 was similar to the

alloy's conventional heat treatment so that the only variable between the data developed under this program and previously developed data for conventionally carburized material would be the carburizing method and temperature. In addition, implementation of the process into industry could be more easily accomplished.

A typical vacuum carburize test cycle procedure utilizing the 1,900°F vacuum carburize temperature is given below:

- a) Place test slugs in VSQ vacuum carburize furnace.
- b) Heat to 1,900°F.
- c) Soak @ 1,900°F for 30 minutes.
- d) Carburize at 1,900°F for 60 minutes, 250 Torr -75 SCF/HR; Target ECD of 0.070-0.110 inch.
- e) Diffuse at 1,900°F for 210 minutes.
- f) Gas quench.
- g) Reheat to 1,550°F.
- h) Soak at 1,550°F for 40 minutes.
- i) 011 guench.
- j) Freeze at -120°F for 120 minutes
- k) Double temper at 300°F for 2 hours each.

Test slugs were exposed to this cycle and then metallurgically evaluated. Results of the evaluation are as follows:

Property	Result		
Uniformity	Acceptable		
Effective Case Depth	0.080 in.		
Surface Hardness	R/C 65		
Case Hardness	R/C 41.0		
Surface Carbon Content	N/A		

The above data is graphically illustrated in Figure 18. Case and core microstructures are shown in Figures 19 and 20. The case microstructure was not considered typical compared to that observed from conventionally





Figure 19. Case microstructure of vacuum 9310 carburized at 1,900°F. Structure consists of large tempered martensite needles, retained austenite, and some carbides.



Figure 20. <u>Core microstructure of 9310 vacuum carburized</u> at 1,900°F. Structure consists primarily of tempered martensite.

carburized material. Very little carbide was observed, large martensite needles were apparent, and retained austenite was prevalent throughout the case structure. The core structure consisted primarily of tempered martensite, which is typical for this alloy. It was not until testing of the geared roller test specimens began that the detrimental effects of the case microstructure were characterized.

After evaluation of the test slugs, the following cycle was utilized for the geared roller test specimens. These specimens required an 0.065-0.085 inch effective case depth after grinding.

- a) Place test slugs, test parts, etc., in VSQ vacuum carburize furnace.
- b) Evacuate chamber to 500 Mu level.
- c) Carburize at 1,900°F for 60 minutes, 250 Torr -75 SCF; Target ECD 0.070-0.110 inch.
- d) Diffuse at 1,900°F for 210 minutes.
- e) Gas guench.
- f) Reheat to 1,550°F hold 40 minutes.
- g) 011 quench.
- h) Cool to -120°F hold 2 hours.
- i) Double temper at 300°F for 2 hours each.

Evaluation of the test slugs ran with these specimens revealed the following:

Property	Carburized @ Summit	Carburized @ Hayes		
Uniformity	Acceptable	Acceptable	_	
Effective Case Depth	0.088-in.	0.0 93 -1n.		
Surface Hardness	R/C 63.0	R/C 63.0		
Surface Carbon Content	0.9-1.2	1.0-1.2		

*Results from three test slugs evaluated at Boeing Helicopters.

Metallurgical evaluation of the carburized case microstructure again indicated an unusually large martensite needle formation. A lack of precipitated carbide and approximately 20 percent retained austenite (determined by X-ray analysis) was also noted. The structure was considered to be the result of (1) carburizing at 1,900°F, (2) an ineffective quench procedure during hardening of the material, and (3) a lack of stress relief annealing in the carbide precipitation range.

As a result, changes to the carburizing procedure were incorporated for items two and three above. The carburizing temperature of 1,900°F was not changed. It was noted that the core microstructure was considered acceptable on all test samples evaluated.

Processing of the single tooth bending fatigue test and rotating fatigue test gears followed closely, from a scheduling standpoint, to the geared roller test specimens. While vacuum carburizing these components at 1,900°F at C.I. Hayes, an inadvertent vacuum furnace temperature overrun occurred whereby the temperature of the furnace exceeded the melting point of copper (approximately 1,980°F). All of the gears had been previously copper plated for carburizing stop-off purposes in the web section, and as a consequence of the over temperature, the copper plate melted on all of the components in the furnace. C.I. Hayes indicated the furnace had most likely reached 2000F before the over temperature was observed. Since the gears were suspended on a bar in the furnace, the molten copper dripped down onto the teeth. In some cases the copper diffused into the case microstructure, as shown in Figures 21 through 24. As a result of the copper melting onto the parts, the effective case depths on the gears varied significantly depending on whether the gear teeth were up or down while on the bar in the furnace. This is illustrated in Figures 25 and 26.



Figure 21. Rotating fatique test gear (left) and single-tooth fatique test gear (right) vacuum carburized in a lot which was inadvertently exposed to 2,000°F. Center web sections were copper plated, the teeth were not.



Figure 22. <u>Magnified view of test gear shown in Figure 21.</u> Copper is seen on tooth tip (arrow).



Figure 23. <u>Flank area of gear tooth shown in Figure 5.9.</u> <u>Arrow indicates copper on tooth surface.</u>

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Figure 24. Root area of gear tooth shown in Figure 22. Arrow shows copper penetrating into the 9310 substrate



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Although these gears had to be scrapped, several important carburizing considerations were brought to light:

- Carburizing at 1,900°F of parts which are copper plated requires precise temperature control to insure the 1,980°F copper meiting point is not achieved. This could pose material and processing restrictions to industrial use of the process.
- Vacuum carburizing furnaces must have temperature overrun controlling thermocouples which are placed relatively close to the production load of components.

While all of the above was occurring, testing of the geared roller test specimens, which were vacuum carburized at 1,900°F, had begun. Approximately 25 of the 60 specimens were tested. Evaluation by Weibull plot analysis of the data revealed rolling contact fatigue lives much less than expected. A brief summary of this data compared with previously obtained Weibull plots from conventionally carburized material is shown in Table 9.

TABLE 9. ROLLING CONTACT FATIGUE LIVES (x 10⁵ CYCLES) OF 9310 VACUUM CARBURIZED AT 1.900°F

Carburized Method	L)	
	B10	B50	890
1,900°F Vacuum Carburize Conventional Carburize	41.9 101.0	72.1 182.0	112 3 250 D

The above data indicated that the 1,900°F vacuum carburized samples exhibited less than one-half the life of conventionally carburized material. Metallurgical evaluation of the vacuum carburized samples revealed unacceptable microstructural characteristics similar to those indicated previously in this report for test slugs. It was noted that even with the manges that were incorporated, a refinement of the microstructure ard an increase in the amount of carbide did not occur

As a result of these gear roller evaluations, processing of the single tooth bending fatigue and rotating fatigue test gears was stopped. Although the

gears had been heat treated, finish grinding was stopped. Due to the problems experienced, it was decided to lower the vacuum carburizing temperature from 1,900°F to 1,800°F, remanufacture, and retest new specimens, i.e., geared roller, single toth bending fatigue, and rotating fatigue test gears.

It should be noted that all of these remanufactured samples were produced within the budget of the original program. Summit Gear, Stulen Machine Co., and Litton Precision Gear are to be commended for their assistance in this regard.

The revised vacuum carburization procedure used for these new specimens was as follows:

- a) Evacuate chamber to 500 Mu.
- b) Soak @ 1,800°F for 35 minutes.
- c) Carburize at 1,800°F for 45 minutes; Target ECD 0.040-0.060 inch.
- d) Diffuse at 1,800°F, 80 minutes.
- e) Gas guench.
- f) Anneal at 1,275°F in nitrogen for 120 minutes.
- g) Soak at 1,525°F for 40 minutes.
- h) 011 quench.
- i) Freeze at -120°F, 3 hours minimum at temperature.
- j) Double temper at 300°F for 2 hours each.

Metallurgical evaluation of test slugs from this cycle showed a refined grain structure in the case with some precipitated carbide at the surface. This structure was a significant improvement over that of the 1,900°F vacuum carburi ed material and indicated that the lower carburizing temperature should produce the desired surface contact fatigue properties.

As a result, the following heat treat cycle was used to vacuum carburize the replacement 3310 geared roller test specimens, Part Number SK²0895-4.

 a clace first slugs, test part, etc., in VSQ vacuum carburize furnace.

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- b) Evacuate chamber to 500 Mu vacuum level.
 (The heaters in the VSQ furnace do not come on until a vacuum of 500 Mu is reached).
- c) Heat to 1,800°F.
- d) Soak @ 1,800°F for 35 minutes.
- e) Carburize 150 minutes at 1,800°F CH₄ 250 Torr -75 SCF/HR; Target ECD 0.070-0.110 inch.
- f) Diffuse for 275 minutes at 1,800°F.
- g) Gas guench.
- h) Stress relieve at 1,250°F for 2 hours in vacuum.
- i) Gas quench.
- j) Reheat to 1,525°F for 40 minutes in vacuum.
- k) 0il quench.
- 1) Deep freeze at -120°F for 3 hours at temperature.
- m) Double temper at 300°F for 2 hours each.

Evaluation of test slugs from these runs revealed the following:

Property	Result
Uniformity	Acceptable
Effective Case Depth	0.082-in.
Surface Hardness	R/C 64.5
Surface Carbon Content	N/A

The case and core microstructures produced in 9310 by this cycle are seen in Figures 27 and 28. The case microstructure consisted primarily of tempered martensite with precipitated carbide at the surface. Some large martensite needles can be seen. The core microstructure appeared to be typical of that of conventionally processed 9310 steel.

Similar to that mentioned previously for X2M, a significant time savings of approximately 60 percent was obtained by vacuum carburizing (approximately 9 hours) compared to conventional carburizing (approximately 24-30 hours). This time savings can also be viewed on the basis that the total time for vacuum



Figure 27. <u>Case microstructure of 9310 steel vacuum</u> carburized at 1,900°F. The structure is primarily martensite with some precipitated carbide at the surface.



Figure 28. <u>Core microstructure of 9310 vacuum carburized</u> <u>at 1,800°F. Structure consists of tempered</u> <u>martensite.</u>

carburization and heat treatment was 21 hours compared to the 24-30 hours required solely for conventional carburization.

Excepting the changes to the carburizing and diffusion times as shown below, a'l of the remanufactured components were heat treated in a similar manner as detailed above. Summit Gear Company heat treated all of these components since time did not permit manufacturing and shipping of two groups of specimens. Also detailed below is the number of specimens in each group that were processed. The results of the testing of these gears are discussed in the next paragraph 5.1.3.

TABLE 10. TEST GEARS MANUFACTURED FROM AISI 9310 VACUUM CARBURIZED AT 1,800°F

Specimen Type	P/N	Number of Specimens	Carburizing Time (min)	Diffusion Time (min)	
Geared Roller	SK20895-4	15	150	275	
Single Tooth	SK29572-10	8	45	80	
Bending Fatigue					
Rotating Fatigue	SK29571-10	20	45	80	

5.1.3 Geared Roller Test Results for X2M and 9310 - A typical Caterpillar geared roller test machine, shown in Figure 29, was used for these tests. Two machines were used. A typical one inch diameter test roll made from the Vasco X2M steel and the mating five inch diameter slave roll can be seen in Figure 30.

5.1.3.1 Gear Roller Test Machine Problems - During the testing, several problems developed with the test machine apparatus which invalidated certain test data. These problems are briefly listed below:

 (a) Lubricating jets in the test machine were clogged with debris, thus not allowing proper lubrication of the test specimen. To remedy this, the machine was completely drained of oil, cleaned and



Figure 29 Gear roller test machine.



Figure 30. Test and slave roll for geared roller testing.

refilled. All oil lines were freed by forced air cleaning, after which testing again commenced.

- (b) Several of the test rolls slipped during testing. This slipping resulted in a greater number of recorded test cycles than the specimen actually experienced. This slipping was caused by dimensional discrepancies on the test roll or in the test fixture. In an effort to eliminate this problem, the following were tried:
 - Increasing the dimensions on the stub end of the test specimens during machining (this was done prior to manufacture of certain test rolls).
 - (2) Flating the stub end of the specimens with copper, nickel, etc.
 - (3) Applying 3M Loctite material to joint.

Items (2) and (3) were relatively unsuccessful.

- (c) Failure, by spalling fatigue, of the five-inch diameter slave roll. Since this component mates with the test roll, spalling of the slave roll results in distress on the test roll.
- (d) Specimen holder test machine shafts spalled. To solve this problem, the shafts were ground down to sub-size and oversize bearings were purchased and installed on the sub-size shaft diameter. Appendix D details the remachining requirements of the test machine shafts.
- (e) Prior to any testing, one 5 inch diameter slave roll had to be reground for each of the original 120 roll test specimens. These were reground since they had previously been utilized on the program detailed in Reference 4. A drawing of the specimen is given in Appendix H. Each slave roll, after regrind, was magnetic particle inspected. Of the 120 slave rolls, approximately one third were

found to contain cracks. None of the rolls with cracks were used in this program.

Testing of all geared roller test samples was not accomplished primarily due to problems (b) and (c) discussed above. However, sufficient data was obtained to provide the significant trends discussed below for 9310 and X2M.

5.1.3.2 Testing and Evaluation - Initial geared roller testing was performed at several hertz compressive stress loadings. The stress varied from 288,000 psi H_Z to 637,000 psi H_Z . Due to the scatter in the test results and the presence of scuffing-type failures rather than pitting or spalling, it was decided that all testing would be accomplished at only one load level of 450,000 psi Hertz stress. (A scuffing mode of failure is one in which heat exchange occurs after an oil film breakdown. Pitting or spalling is a fatigue failure with no evidence of heat or oil film breakdown at the critically stressed contact point.)

Each test roll was labeled with mill heat number, heat treat vendor and vacuum carburized temperature. This labeling, along with the number of rolls manufactured and tested, is shown in Table 11. The complete test data obtained for the 9310 specimens is also shown.

The method of evaluating surface contact fatigue data is by Weibull analysis, i.e., Weibull Distribution. This method utilizes straight line graphs to represent cumulative percentages of failures by means of appropriate coordinate scales. This method is especially convenient for engineering purposes because a clear graphic picture of fatigue life distribution enhances conveyance of statistical decisions based on the data. Reference 3 lists the method of evaluating data by this technique.

TEST ROLL LABE	L MATE	RIAL	MILL HEAT NUMBER	VAC CARB TEMP°(F)	VAC CARB VENDOR	NUMBER OF ROLLS MANUFACTURED	NUMBER OF ROLLS TESTED
-1A	Vasco	X2M	5842	1,800	Summit	15	10
-18	Vasco	X2M	5842	1,800	Hayes	15	10
-2A	Vasco	X2M	86510	1,800	Summit	15	10
-28	Vasco	X2M	86510	1,800	Hayes	15	10
- 3A	931 0		86510	1,900	Summit	15	5
- 38	9310		86670	1,900	Hayes	15	5
- 4A	931 0		86043	1,900	Summit	15	7
-4B	9310		8 6 043	1,900	Hayes	15	6
-4	9310		86043	1,800	Summit	15	10

TABLE 11. GEARED ROLLER TEST SPECIMEN IDENTIFICATION

Each test group was plotted individually on Weisull Distribution paper. In addition, the data was plotted together in one sample size. This was done since the combining of similar groups result in more significant data because, as the sample size increases, the reliability of the data also increases. The data obtained is shown in Figures 31 to 33. The raw data is shown in Appendix P.

One of the values of the Weibull distribution plot is that comparisons can be made by determining the variation of Bl0, B50, and B90 life of the different



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Figure 31. Weibull plot of gear roller test data for AISI 9310 vacuum carburized at 1,900°F.



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Figure 32. <u>Weibull plot of gear roller test data for AISI 9310 vacuum carburized</u> <u>at 1,800°F.</u>



Figure 33. Weibull plot of gear roller test data for AISI 9310 and Vasco X2M.

groups tested. Bl0 life is defined as the life, in cycles, at which 10 percent of all tested samples have failed. B50 and B90 lives are similarly defined.

After determining the B10, B50, and B90 lives of 9310 and X2M, bar charts comparing the vacuum carburized versus conventionally carburized rolling contact fatigue lives were developed, as shown in Figure 34. The conventionally carburized data was obtained from References 4 and 5.

Evaluation of the data revealed the following:

9310 Steel

Comparison of the vacuum carburized 9310 steel, carburized at $1,800^{\circ}$ F, with the conventional carburized data revealed the 810 and 850 lives of the former were slightly lower than the latter, as seen in Table 12. B9C lives were the same. This data is also shown in Figure 34.

TABLE 12. 9310 ROLLING CONTACT FATIGUE LIFE COMPARISON MATERIAL VACUUM CARBURIZED AT 1800F.

Carburized Method	<u> </u>		
	B 10	B50	B9 0
Vacuum	6.9x10 ^s	1.55×10 ^e	2.55×10 ⁸
Conventional	1.17×10 ⁶	1.85×10 ^e	2.55x10 ⁶

Also illustrated in Figure 34 are the lives of the 9310 vacuum carburized at 1,900°F. The lives of material produced at the 1,900°F vacuum carburizing temperature were much reduced compared to the lives of conventionally carburized steel.

Observing Figure 31, little effect of the two different mill heats or vacuum carburizing vendors is apparent. This is one of the reasons that, when redeveloping the heat treatment of this material at 1.800° F, it was decided to



Figure 34. <u>Comparison of rolling contact fatigue lives of conventional vs vacuum</u> <u>carburized steel.</u>

use only one mill heat and one vacuum carburizing vendor (other salient reasons were cost and schedule).

Vasco X2M

The results of the X2M geared roller tests are compared in Figure 34 with those obtained previously for conventionally carburized X2M. The results show an improvement in rolling and sliding performance of the alloy based on B10. B50, and B90 lives compared to those of conventionally carburized X2M.

During these geared roller tests, the single tooth bending fatigue and scoring tests were initiated. As will be discussed in paragraphs 5.1.4 and 5.1.5 covering the results of these tests, an improvement in properties similar to those shown in Figure 34 was not obtained. The reason for the variability in property improvements was attributed to the interconnected carbide network in the case of X2M. The effects of this microstructure are discussed further in the following section.

5.1.3.3 Metallurgical Evaluation of Test Samples

Several test rollers of each alloy were metallurgically evaluated. The following metallurgical tests and characteristics were evaluated and documented:

- (a) Spall circumferential and axial length dimensions
- (b) Temper etch for grinding burrs
- (c) Surface hardness
- (d) Core hardness
- (e) Effective case depth determined at R/C 50 depth
- (f) Case and core microstructure.

Metallurgical evaluation of a representative sampling of the geared roller test specimens is given in Table 13 with corresponding microstructures from test specimens shown in Figures 35 through 37.



Figure 35. Vasco X2M geared roller test specimen.



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Figure 36. Case microstructure of X2M geared roller test specimen



Figure 37. Case microstructure of X2M geared roller test specimen

		Spall Dimension	Surface	Core		
Material	Specimen Number	Circumferential Length-in.	Axial Length-in,	Hardness R/C	ECD (IN)	Hardness R/C
Vasco X2M	-1A,1	0.17	0.12	63.5	0.080	43.5
Vasco X2M	-18,5	0.24	0.20	62.0	0.092	45.0
Vasco X2M	-2A,6	0.15	0.12	61.0	0.081	44.0
Vasco X2M	-28,1	0.18	0.06	63.5	0.078	43.0
9310	-4,10	0.20	0.29	63.5	0.065	38.5

TABLE 13. METALLURGICAL DATA OF GEARED ROLLER TEST SPECIMENS

As mentioned in paragraph 5.1.1, microstructural evaluation of the Vasco X2M material disclosed that the case consisted of tempered martensite with a continuous carbide network. This carbide network was found to extend to a maximum depth of 0.020-in. Data detailed in Reference 6 indicates that the carbide network has been shown to produce better contact fatigue life than that produced by a pure martensite structure. Apparently, the network acts as a stiffener to the structure. This is consistent with the results shown in this work since the vacuum carburized AISI 9310 material case microstructure did not contain a carbide network and did not exhibit an equivalent rolling contact fatigue life. As the data bears out, the vacuum carburized Vasco X2m contact fatigue life was superior to that of the 9310 steel. It is noted here that the X2M case microstructure was not typical of that observed in conventionally carburized transmission components due to carbide networks that are not permitted by specification. As mentioned previously, refinements to the vacuum carburizing technique for X2M were undertaken in Phase IV of this program to eliminate this type of unwanted structure.

The core microstructure of both materials, Vasco X2M and 9310, was considered typical to that observed during conventional heat treatment. The only difference noted was that there was less than one percent visual estimate free ferrite in the core microstructure of the Vasco X2M materials. This, however, is not considered significant.

Evaluation of several of the test rollers also confirmed that the carburization was uniform in both alloys with no evidence of spottiness. Surface hardness, case hardness and effective case depths were, for the most part, within drawing requirements. Dimensional evaluation of the spall present on each failed specimen revealed all to be similar. Representative examples of spall dimensions were given in Table 7.

5.1.4 Single Tooth Bending Fatigue Life Test Results

The results of each alloy are discussed separately below.

5.1.4.1 AISI 9310 Data Evaluation - All 9310 test gears in this program were manufactured from a single heat of double vacuum melted material. The results of this testing are presented in the S/N curve shown in Figure 38. All of the raw data are shown in Appendix Q. Baseline data for air melt quality and double vacuum melted quality 9310 were used for comparison to this data. Figure 39 shows the results of the previous double vacuum melt 9310 testing. Except for the fact that the 99% confidence band on the mean for the vacuum carburized material is slightly smaller than that for the conventionally carburized material, the two curves exhibit no significant differences. Based on this data, it must be concluded that the vacuum carburizing process did not affect the bending fatigue strength of the AISI 9310 material in any way, either beneficially or detrimentally.

An interesting sidelight to the vacuum vs. conventional carburization process evaluation is to compare both Figures 38 and 39 to Figure 40 which shows similar bending fatigue data for air melt 9310 steel. The improvement obtained from the double vacuum melt over air melt material is quite obvious and very significant.

5.1.4.2 Vasco X2M Data Evaluation - Two different heats of Vasco X2M material were used in manufacturing these test gears. All of the gears were machined by a single vendor. However, each heat was separated into two groups, and each group was heat treated by a different heat treat vendor. The different heats are identified by the dash number associated with each part number while the heat treaters are identified as "Heat Treater A" and "Heat Treater B,"


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Figure 38 Single tooth bending fatigue life data of conventionally carburized DVM 9310.



Figure 39. <u>Single tooth bending fatique life data of conventionally carburized</u> <u>DVM 9310. Data from Boeing Helicopters Test Report MF-370</u>



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respectively. By manufacturing the gears in this manner, it was possible to determine if there was a significant difference in the results between either different heats of the same material or between different heat treaters working to the same heat treatment specifications.

Figure 41 shows all of the Vasco X2M data plotted together, i.e both heats and both treaters. From Figures 42 and 43 which show the results for the two heats of material separately, it can be concluded that there does not appear to be a difference between the two heats of material used. However, Figures 44 and 45 indicate that there is a difference between the two heat treat vendors, with Vendor B producing gears with a significantly higher bending fatigue load capacity.

The major consideration in this program, however, was whether or not the vacuum carburizing procedure significantly affected the bending fatigue capacity of the gears when compared with that of conventionally carburized material. As can be seen by comparing the data in Figure 46, which is the existing Vasco X2M baseline data, with any of the foregoing charts, the vacuum carburized Vasco >2M gears have significantly lower bending fatigue capacity, regardless of heat or heat treat vendor.

As mentioned previously in paragraph 5.1.1, metallurgical evaluation of the vacuum carburized X2M material indicated the presence of a heavy carbide network in the surface and near surface microstructures. Since this is an undesirable condition for good bending fatigue resistance, the heat treatment procedure was revised in an effort to improve this property. The revision of the X2M vacuum carburization procedure was conduced in Phase IV, the results of which are discussed in paragraph 5.4.

5.1.5 Scoring Test Results

The data obtained from the scoring tests was evaluated in light of previous data obtained from similar test programs, References 5, 6, and 7. All three of the referenced programs used the same test rig and test method as the current program. The basic geometry of the test gears used in the Reference 6 and 7 testing was identical to the current specimen geometry. However,





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MIL-L-7808 oil was used as the test oil. The testing defined in Reference 7 was conducted using gears of similar design and manufacture as those used in the current program but of somewhat large size (i.e., > 0.25-inch face width and 12.75 pitch diameter gear set operating on 10 inch center distance) than the current gears. The 9310 material used in Reference 2 was for air melt material and that used in Reference 4 was for single vacuum melt material, 0.24 carbon.

The raw data obtained from the scoring tests is shown in Appendix R. Table 14 presents a statistical summary of the data obtained in the current program and, for reference purposes, the same data from the baseline testing (References 5, 6, and 7). Figure 47 shows the same data graphically. Several very significant facts are immediately apparent from examination of this data. The most significant is that the mean flash temperature at failure is higher for the vacuum carburized 9310 and X2M materials than it is for the baseline data. Also important is that for X2M, the vacuum carburizing process (i.e., heat treater A vs. B) did result in a performance variation, but no difference based on mill heat (-1 vs -2) within each heat treater. The coefficient of variation (which is the standard deviation divided by the mean) is guite low for all samples.

In general, the data from the current program is slightly more consistent (within parameters) with less scatter, as evidenced by the smaller standard deviations, than the baseline data.

In addition, it is interesting to note that the scoring load capacity of gears heat treated by Vendor A is higher than that of gears heat treated by Vendor B, while the relationship for bending fatigue load capacity is exactly the opposite (i.e., for bending, Vendor B heat treat resulted in higher load capacity than Vendor A). Both conditions are likely due to the heavy continuous carbide networks which were present in gears heat treated by Vendor A. Since the carbide network condition reduces bending capacity but improves both durability and scoring, these results are certainly consistent.

RESULTS
TEST
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TABLE

PART NUMBER: MIIL HEAT	SK2957	1-1	SK295	71-2	SK29571-10
	TELED	YNE	CARTI	ECH	CARTECH
MATERIAL:	VASCO	Х2М	VASCO	X2M	AISI 9310
HEAT TREATER:	A SUMMIT	B HAYES	A Summert	B HAYES	A SUMMIT
FLASH TEMPERATURE AT FAILURE, ^O F					
MEAN	532.9	480.8	525.9	0.692	420.5
STANDARD DEVIATION	16.35	15.02	12.89	17.51	16.27
COEFFICIENT OF VARIATION	160.0	10.031	0.025	0.037	60.0
BASELINE DATA					
(REF. 3, 4, 5 DATA BASE)				-	
F. VSH TEMPERATURE AT FAILURE, ^O F					
MEAN		451	6.8	-	354.6
STANDARD DEVIATION		2	8.58		18.39
COEFFICIENT OF VARIATION		С	.063		0.052



5.2 PHASE II - SURFACE DURABILITY EVALUATIONS

In Phase II, the surface durability of the vacuum carburized 9310 material was investigated through testing and evaluation of spur and spiral bevel gears. The surface durability of X2M was not evaluated because the vacuum carburization procedure was not optimized, as discussed in Section 3 and paragraph 5.4.

5.2.1 Introduction

The primary concern of the transmission gear design engineer is to provide adequate tooth strength. For this reason, year materials are selected mainly for beam strength. To improve the sliding behavior of the material, the usual procedure is to then harden the surfaces. Due to the emphasis on the development of gear materials for strength rather than for good sliding characteristics, most aircraft gears will pit or spall under conditions which are far less severe than those which would cause tocth breakage, particularly at low speeds. In many aircraft gear applications, pitting is a limiting factor in reducing the size and weight of transmissions.

As was the case with the previously described bending and scoring tests, the surface durability or pitting resistance of gears heat treated by the vacuum process must be evaluated to ensure that these gears exhibit load capacity levels at least equivalent to those of conventionally carburized gears.

Because the vacuum carburizing process was not optimized for the X2M material at the time the durability test gears were manufactured, no X2M gears were tested.

5.2.2 Results and Analysis of 9310 Gear Tests

Initial light scoring was observed at the 100 percent load level on most of the spur gear test specimens. However, after the gears were lightly polished, the light scoring appeared to stabilize and "heal over" at the conclusion of the 100-percent load run.

Initial light scoring is a state of lubrication phenomena which develops as a result of surface asperity contacts. The local high spots concentrate the

load in these areas permitting metal-to-metal contact, along with concentrated pressures. Continued operation at moderate loads will eventually wear down the localized high spots (asperities) and thereby permit improved load distribution which in turn will result in a healed over (polished) condition.

A frosting condition on the pinion dedendum (the area between the pitch circle and the start of the root radius) was usually observed at the 160 percent load level; however, the severity of this condition was not apparent during normal visual observation.

The pitting condition sustained by the gear test specimens appears to be typical of destructive pitting usually found in the dedendum of the driving member in a reduction gear drive. This condition is characterized by the appearance of pits, of at least 1/32-inch in diameter, in the dedendum region. This type of pitting failure will usually progress in size and number of pits, with repeated stress cycles.

The data obtained from the durability testing was statistically analyzed in the same manner as the single tooth bending fatigue data. The results of this analysis are shown in Figures 48 and 49 for the spur and bevel gears, respectively.

When this data is compared to the spur gear durability baseline data shown in Figure 50, it is obvious that the vacuum carburizing process has not adversely affected the surface load capacity of the gears. It should be noted that the baseline data was obtained with air melt material and thus some improvement may be expected simply through the use of double vacuum melt material in the vacuum carburizing program. Even considering this effect, however, the comparison indicates that the vacuum process produces gears with surface load capacity characteristics at least as good as those of conventionally carburized gears. In all likelihuod, the characteristics of the vacuum carburized gears is better in this respect.

In reviewing the spiral bevel gear data, it should be noted that no actual pitting failures occurred. Due to the very high loads which were required in this program, fretting occurred at the interface between the pinion hore and









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its supporting shaft. This fretting eventually generated a crack at the bore-shaft interface and the tests were halted each time a crack became apparent. Several attempts were made to improve this joint. The attempts included higher bolt torques and improved finish at the joint. These changes were only successful in forestalling the occurrence of the cracks, they did not eliminate them. Despite this problem, however, enough cycles were run without a pitting failure to provide a worst case indication of the behavior of the material. In performing the analysis of the bevel gears, therefore, each test data point was treated as a durability failure even though no pitting occurred. In this way, the data provided an indication of the behavior of the vacuum carburized gears.

5.3 Phase III - VACUUM CARBURIZED GEAR TESTING AND IMPLEMENTATION PLAN

The objective of this phase was two fold: first, to vacuum carburize and heat treat a gear of complex geometry and test it in a Boeing Helicopters CH-47C transmission, and second, to develop a plan to implement the vacuum carburization procedure developed in this program into production. The following sections detail the results of these efforts.

5.3.1 Vacuum Carburized Gear Testing

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The gear selected for vacuum carburizing evaluation and helicopter transmission testing was a spiral bevel input pinion gear, Part Number 11405245-10, Serial Number M5373, and is shown in Figure 51. The drawing for this part is found in Appendix S. This gear is used in Boeing Helicopters CH-47C combining transmissions, and is located in the position shown in Figure 52. This gear was selected due to its complex geometry and the extensive amount of time required for conventional carburization and hardening, which could be substantially reduced by use of the vacuum carburization and heat treat procedure developed in this program.

This gear was vacuum carburized by Summit Gear and heat treated and machined by Litton Precision Gear. The carburization and heat treat processing cycle as well as other processing details are discussed in paragraph 4.2.3.







Figure 52. Location of spiral bevel input pinion gear in CH-47C transmission.

Following processing, the gear was assembled in a CH-47C combining transmission and bench patterned in accordance with normal Boeing Helicopters procedures. The bevel gear contact patterns obtained were typical of normal production. All components used in the gear box were standard production parts except that the right hand engine drive pinion was the vacuum carburized test part.

After the gear box was assembled, it was installed in the production test stand and subjected to a standard production run-in procedure. This procedure consists of running the box, in sequence, at 10 percent load and full speed for one-half hour, at 50 percent load and full speed for one-half hour, and at 100 percent load and full speed for one hour. The load run-in is followed by a visual inspection of the gear tooth contact patterns by removing only the pinion cartridge from the housing.

Subsequent to the production run-in, the gear box was reinstalled in the test stand and run for 20 hours at 100 percent of the single engine power and speed rating. During this running, all standard instrumentation (measuring oil temperature and pressure, ambient air temperature, shaft torque, etc.) was operational.

The gear box was then removed from the test cell, completely disassembled, and visually inspected. The condition of the vacuum carburized gear and all other components was typical of that observed on other gear boxes after similar loading conditions. No distress of any kind was observed, and all parts were in acceptable condition.

Metallurgical evaluations (both destructive and nondestructive) were conducted on the gear after the test. These evaluations included the following: magnetic particle inspection; temper etch inspection (to detect grinding burns); chemical composition; case and core microstructures; effective case depths and surface and core hardnesses; dimensional checks of the gear teeth root fillets; carbon content of the carburized case as a function of depth from the gear tooth surface; and a residual stress profile of a gear tooth. All of these items conformed to the requirements of the engineering drawing and related specifications. No cracks or temper burns were found during the magnetic particle or temper etch inspections, respectively. The chemical composition conformed to the requirements of Boeing Helicopters Specification BMS 7-249C, and the carbon content as a function of distance from a tooth flank surface was similar to that for a conventionally carburized gear. The residual stress profile was also similar to that for a conventionally carburized and shot peened gear. The case and core microstructures, shown in Figures 53 and 54 were also similar to those of a conventionally carburized gear. A closer view of the vacuum carburized gear teeth is shown in Figure 55. The appearance of the teeth surfaces is the same as that of a conventionally carburized gear. The evaluations are listed in the Boeing Helicopters Materials Engineering Laboratory Report number 88-164 found in Appendix T.

5.3.2 Implementation Plan

Implementation of vacuum carburization into the Aerospace gear industry could be relatively easily accomplished. As discussed previously in this report, the two spiral bevel gears produced in Phase III were manufactured alongside other gears of the same type, with the only exception being the use of a vacuum carburization procedure in place of a conventional carburization procedure. It should be noted that vacuum carburization was not conducted in a laboratory, but in an actual production environment at Summit Gear Corp. Use of this vacuum carburization procedure significantly reduced the amount of time needed to produce these two gears compared to that required to produce those in the same group that were conventionally carburized. All other production processes, such as grinding, heat treatment, and inspection, were the same for all gears.

Six items have been identified as requiring modification to allow the use of vacuum carburization in production. Each is discussed separately below.

 Develop a vacuum carburization specification for AISI 9310. Boeing Helicopters carburizing and hardening specification for 9310, 0210-12023-1, would be ammended to include the vacuum process as a second carburization method. To distinguish between the two carburization methods, the vacuum process would be identified as Type II, and the conventional process would be identified as Type I.



Figure 53. Case microstructure of vacuum carburized 9310 spiral bevel gear. Structure is primarily tempered martensite.



Figure 54. <u>Core microstructure of vacuum carburized 9310</u> <u>spiral bevel gear.</u> <u>Structure is primarily</u> <u>tempered martensite</u>.



Figure 55. Vacuum carburized 9310 gear tooth surface.

All of the the details of the process including times and temperatures; gas type, pressure, and flow rates; etc., would be clearly specified.

- Alter part drawings to indicate that the part is to be manufactured by vacuum carburization. The drawing note concerning carburization method would be changed to reflect the vacuum carburization method, as mentioned in number 1 above.
- 3. Alter manufacturing plans to change the carburization procedure from the conventional method to the vacuum method. The manufacturing plan for each part/vendor would be changed to detail how each would specifically produce the gear. Included would be all times and temperatures; gas type, pressure, and flow rates; and all other details of the method.

In addition, manufacturing plans may be altered to allow less stock removal during final grinding, thus further decreasing gear production time. Any change in grinding stock would only be allowed, however, after careful evaluation of the amount of part warpage that occurs during vacuum carburization compared to that which occurs during conventional carburization. If it is found that the shorter vacuum carburization time does not allow for as much part warpage as the longer conventional caburization method, then a reduction in the grinding stock may be allowed.

- 4. Vendors and related equipment must be qualified. This is a standard procedure at Boeing Helicopters when any new vendor or equipment is used to produce parts. The vendor would be surveyed and all production processes would be audited to ensure that each conformed to the applicable Boeing Helicopters specifications.
- Initially process small gears with simple geometries to ensure process stability. Once the process has been found to be stable at a particular vendor, more complex, critical gears would be produced.

6. Conduct a metallurgical evaluation/qualification of one part of each part type that is vacuum carburized. This is a standard procedure at Boeing Helicopters and is done when any significant process change has occurred or a new vendor is producing parts.

An ongoing evaluation of the performance of vacuum carburized gears would also be conducted through an examination of parts brought in for routine overhaul. Selected parts would be traced and the performance compared to that obtained from conventionally carburized parts of the same type with a similar service history and time.

All other items associated with the production of these gears, such as heat treatment, machining, grinding, surface treatments, etc., would remain unchanged.

Boeing Helicopters has a great interest in furthering the development/use of vacuum carburization, and suggests that the implementation plan discussed above be funded. This topic is discussed in the Recommendations Section 7.

5.4 PHASE IV - FURTHER DEVELOPMENT OF THE VACUUM CARBURIZATION PROCEDURE FOR VASCO X2M

As previously mentioned in Sections II and 5.4.1 of this report, this fourth phase was added to the program during the Phase I effort so that optimization of the vacuum carburization procedure for the Vasco X2M material could be continued in parallel with the gear testing of the 9310 material. During the originally planned Phase I effort, the vacuum carburization procedure was optimized for 9310, but not for X2M. During this Phase, the valuum carburization procedure for X2M was further developed, but due to cost and scheduling restrictions, was not optimized. Because of this, the majority of the gear and gear element testing that was conducted on 9310 (see paragraphs 5.2 and 5.3) was not conducted on X2M. The following paragraphs describe the results of the additional X2M vacuum carburization evaluations.

Six trial vacuum carburization cycles were conducted in which the times and temperatures were varied to try and develop a vacuum carburization procedure

that would yield an optimum case microstructure. The results of these runs, which were identified as numbers IV-1 through IV-6, are discussed below.

In the first cycle, IV-1, test slugs similar in configuration to those utilized in Phase I were exposed to the following cycle and then metallurgically evaluated:

CYCLE IV-1

- Heat to 1,900°F in a vacuum and hold for 35 minutes to stabilize temperature.
- 2. Cool to 1,800°F and hold for 10 minutes.
- 3. Carburize at 1,800°F for 15 minutes.
- 4. Diffuse at 1,800°F for 60 minutes.
- Gas quench, and hold for 10 minutes, 250 Torr at 36 CFM of methane gas.

The case microstructure resulting from this cycle consisted of tempered martensite with a heavy carbide network located in the grain boundaries, which was not acceptable.

In an attempt to reduce the amount of carbides in the grain boundaries, the following cycle (1V-2) was run.

CYCLE IV-2

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- 1. Heat to 1,900°F and hold for 35 minutes.
- 2. Cool to 1,650°F and hold for 10 minutes.
- Carburize at 1,650°F for 240 minutes, 650 CFM methane/propane, 175 Torr.
- 4. Diffuse at 1,800°F for 60 minutes.
- 5. Nitrogen quench.

The case microstructure of the material from this cycle also contained the carbide network at the grain boundaries similar to that resulting from Cycle IV-1. In the third cycle, IV-3, the test slugs were carburized at a higher temperature and a shorter time than that used in run IV-2, followed by several subcritical anneals, as shown.

CYCLE V-3

- 1. Heat to 1,900°F and hold for 35 minutes.
- 2. Cool to 1,800°F and hold for 10 minutes.
- Carburize at 1,800°F for 15 minutes, 360 CFM methane/propane, 250 Torr.
- 4. Diffuse at 1,800°F for 60 minutes.
- 5. Gas guench.
- 6. Anneal at 1.386°F for 120 minutes.
- 7. Cocl to 1,150°F and hold for 120 minutes.
- 8. Heat to 1,380°F and hold for 120 minutes.
- 9. Cool to 1,150°F and hold for 120 minutes.
- 10. Air cool.

This cycle did not remove the heavy carbide network at the grain boundaries. as seen in Figure 56 and 57. To again try and reduce the carbide network, the carburization temperature was reduced to $1,700^{\circ}$ F, and the carburization time was increased to 60 minutes, resulting in cycle IV-4 shown.

CYCLE IV-4

- 1. Heat to 1,900°F and hold for 35 minutes.
- 2. Curl to 1,700'F and hold for 10 minutes.
- 3. Carburize at 1,700°F for 60 minutes.
- 4. Diffuse at 1,800°F for 15 minutes.
- 5. Gas guench.

Evaluation of the case microstructure resulting from cycle IV-4 revealed a region of martensite needles at the surface of the test slugs with a carbide network in the grain boundaries immediately below this region, as seen in Figure 58.



Figure 56. Case microstructure of X2M resulting from vacuum carburitation run IV-3.



Figure 57. <u>Higher magnification of the microstructure</u> shown in Figure 56.



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Figure 58. Case microstructure of X2M resulting from vacuum carburization run IV-4.

To eliminate the martensite needles and reduce or eliminate the carbide network, the diffusion time was altered and a stress relief and temper was added to create the desired case microstructure. The resultant cycle (IV-5) was as follows:

- 1. Heat to 1,900°F and hold for 35 minutes.
- 2. Cool to 1,700°F and hold for 10 minutes.
- Carburize at 1,700°F for 180 minutes, 75 CFH, 175 Torr, 10 propane +65 methane.
- 4. Diffuse at 1,700°F for 180 minutes.
- 5. Gas guench.
- 6. Stress relieve at 1,150°F for 90 minutes.
- 7. Heat to 1,850°F and hold for 30 minutes.
- 8. 0il quench.
- 9. Freeze to -120 and hold for 120 minutes.
- 10. Double temper at 600 for 120 minutes each.

Evaluation of the microstructure resulting from Cycle IV-5 showed that the martensite needles were eliminated and that the heavy carbide network had been reduced slightly, as seen in Figure 59.

The reduction in the amount of carbide network resulting from cycle IV-5 suggested that increasing the time and/or number of tempers could further reduce or eliminate the carbide network. Incorporation of this idea in the carburization procedure resulted in Cycle IV-6 shown below.

- 1. Heat to 1,900°F and hold for 30 minutes.
- 2. Cool to 1,800°F and hold for 10 minutes.
- 3. Carburize at 1,800°F for 60 minutes.
- 4. Diffuse at 1,800°F for 15 minutes.
- 5. Gas guench.
- 6. Stress relieve at 1,250°F for 90 minutes.
- 7. Heat to 1,850°F and hold for 95 minutes.
- 8. Oil quench.
- 9. Freeze to -120°F and hold for 180 minutes.
- 10. Double temper at 600°F for 120 minutes each.



Figure 59. Case microstructure of X2M resulting from vacuum carburization run IV-5.

11. Reheat to 1,850°F and hold for 45 minutes.

- 12. 011 guench.
- 13. Freeze to -120°F and hold for 180 minutes.
- 14. Double temper at 600°F for 120 minutes each.
- 15. Reheat to 1,850°F and hold for 45 minutes.
- 16. 011 guench.
- 17. Freeze to -120°F and hold for 180 minutes.
- 18. Double temper at 600°F for 120 minutes each.

The case microstructure resulting from Cycle IV-6 is seen in Figure 60. The amount of carbide network in the grain boundaries was reduced compared to that found previously in the case of the Cycle IV-5 material. A hardness profile taken on the test slug at the flank, root, and root fillet showed that an acceptable hardness was obtained at the surface of the part and that the hardness was uniform in these three areas at various depths.

Using Cycle IV-6, single tooth bending fatigue test gears were produced and evaluated. The results of this testing are shown in Figure 61. Comparison of this data with that of X2M with heavy carbide from Phase I shown in Figure 41 indicates that the improved procedure increased the bending fatigue capacity of the material slightly. However, the increase was not great enough so that it was equal to that of baseline material. This data indicated that further development of the vacuum carburization procedure for X2M was necessary to optimize properties. However, cost and scheduling restrictions caused this effort to be discontinued at this point.


Figure 60. <u>Case microstructur of X2M resulting from vacuum</u> carburizațion run IV-6.





6. SUMMARY AND CONCLUSION

6.1 AISI 9310 MATERIAL

- 1. A vacuum carburization procedure was successfully developed and optimized for the AISI 9310 alloy. This was accomplished on test coupons right on through to a production component. The time required to carburize this material was decreased approximately 60 percent. The resultant case microstructure was comparable to that of conventionally carburized material.
- 2. The single tooth bending fatigue life and scoring resistance of the vacuum carburized 9310 material was equivalent to that of conventionally carburized 9310 material. The B90 rolling contact fatigue life of the vacuum carburized 9310 material obtained from gear roller tests was also equivalent to that of conventionally carburized 9310. The B50 and B10 rolling contact fatigue lives of the vacuum carburized 9310 material was slightly less than that of conventionally carburized 9310.
- 3. The surface durability of the vacuum carburized 9310 material was evaluated through spiral bevel and spur gear testing, and was found to be equivalent to that of conventionally carburized material.
- 4. A production spiral bevel input pinion gear for a Boeing Helicopters CH-47C Combining Transmission was vacuum carburized and heat treated according to the processing conditions developed in this program, and then tested in an actual transmission. Post test evaluations of the test data and the gear showed that it performed as well as a conventionally carburized gear. These results indicate that use of the vacuum carburization process developed in this program for 9310 can produce gears in a shorter period of time compared to conventional carburization and that acceptable properties can be obtained from such a gear.
- A plan was developed which will allow the vacuum carburization process to be implemented into the helicopter transmission gear production process.

6.2 VASCO X2M MATERIAL

- A nonoptimized vacuum carburization procedure was developed for the Vasco X2M alloy. The case microstructure resulting from this nonoptimized procedure contained a uniform, continuous carbide network which had deleterious effects on the alloys properties. However, this uniform case microstructure was obtained without the preoxidation procedure which is required to obtain a uniform case microstructure in conventional carburization. Further efforts are required to optimize this process so that acceptable mechanical properties may be obtained.
- The single tooth bending life of the vacuum carburized X2M material was less than that of conventionally carburized X2M. This lower fatigue life was attributed to the unacceptable carbide network in the carburized case.
- 3. Further work is required to continue the development of an optimized vacuum carburization procedure for X2M.

General Comment

The results of the Phase I Vasco X2M gear element testing illustrate the need to fully evaluate all of the mechanical properties of a new alloy and/or process being considered for use in a gearing application. In this program, a thorough examination of all properties, i.e., single tooth bending fatigue life, scoring resistance, as well as geared roller testing, was conducted and revealed that the vacuum carburization process as developed for X2M yielded poor single tooth bending fatigue properties while at the same time yielding good geared roller properties. If only geared roller tests had been conducted and the decision to use the currently developed vacuum carburization process for X2M in production had been based solely on these results, serious problems with transmissions would have occurred.

To avoid the possibility of a material or process being used in critical components without first undergoing thorough testing, it is suggested that a specification be developed in which a minimum amount of testing be required.

These specified tests would ensure that all pertinent properties of the new alloy and/or process are fully understood and that all loading conditions normally encountered by the component be evaluated.

7. RECOMMENDATIONS

It is recommended that a follow on program to optimize the vacuum carburization procedure for the Vasco X2M alloy be funded. This alloy is currently used in large quantities by Boeing Helicopters as well as other companies. The reduction in carburization and heat treatment time would significantly reduce acquisition costs of any products incorporating this alloy. The knowledge gained from this program on vacuum carburization indicates that the procedure can be optimized and implemented into production.

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9. GLOSSARY

- R/C Rockwell Hardness, C Scale
- ECD Effective Case Depth
- AGMA Aerospace Gear Manufacturers Association
- CFH Cubic Feet per Hour (gas flow)
- Torr A unit of pressure equal to 1.316 x10³ atmospheres
- HPSTC Highest Point of Single Tooth Contact

10. APPENDICES

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APPENDIX A CARPENTER TECHNOLOGY TEST CERTIFICATE FOR 9310 MATERIAL

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CARPENTER TECHNOLOGY CORPORATION

CERTIFICATE OF TESTS

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APPENDIX H SCORING FATIGUE TEST GEAR DRAWING NUMBER SK29571





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	APPENDIX I	TELEDYN MATERI/	IE VASCO AN AL	ALYSIS REPORT	OF X2M	
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Your Order	No. 8-4435-5-00	80				
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Brand:	VDH-VAR Vad	co X2 Modifi	ed Per Boeing	Spec BHS 7-223C	Type III Ultraso	aic Inep.
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	Hagnetic Pa	rticle Inspec	:t ios -	T - 7/1 0/0 1 - 7/2 0/0		
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APPENDIX J STATISTICAL ANALYSIS METHOD FOR SINGLE TOOTH BENDING FATIGUE DATA

The data from the single tooth bending fatigue test program was analyzed statistically using the same Boeing Vertol computer program used for the earlier test programs. Since comparisons between the results obtained from the earlier programs and those obtained from this program have been made, it is essential that the analysis techniques be consistent.

The statistical analysis was based on the theory that for a particular material there is a definite bending (tensile) stress which the material will be able to withstand for an infinite number of cycles, without the occurrence of a failure. In other words, as long as the stress in a component is maintained at a level equal to or lower than the endurance limit, a failure will never occur. This is a reasonable approach sine, at this time, all Boeing Vertol transmission gears are designed for "infinite" life (i.e. at overhaul they are only removed from service if a defect is found). On the basis of this theory, it was assumed that the mean stress/life (S-N) curve takes the general form:

$$S = \overline{S} + \frac{\alpha}{N^{Y}}$$
(1)

where: S = stress at failure

- **S** = endurance limit stress
- a = material constant
- Y = test gear specimen configuration constant (shape factor)
- N = cycles at which failure occurs

After an empirical evaluation of the material constants, the stress at any number of cycles can be evaluated, using Equation (5), by calculating the \overline{X} value based on an individual data point. In order to determine the mean ($\overline{5}$) stress for the entire group of data at a particular number of cycles (after the individual \overline{X} value for each data point is calculated at these cycles), the individual \overline{X} values are then surmed up and divided by the number of data points (using Equation 6).

$$\widetilde{\mathbf{X}} = \frac{\mathbf{X}_{\mathrm{F}}}{\begin{bmatrix} 1 + \frac{(\alpha/\overline{S})}{N_{\mathrm{E}}} \end{bmatrix}} \begin{bmatrix} 1 + \frac{\alpha/\overline{S}}{N_{\mathrm{E}}^{\mathrm{Y}}} \end{bmatrix}$$
(5)

$$\overline{S} = \frac{\Sigma \overline{X}}{D}$$
(6)

where: \overline{S} = mean stress for total group of specimens at N_p cycles

 $\overline{\mathbf{X}}$ • stress at N_R cycles based on a particular data point

 $X_{\rm F}$ = stress at failure for a particular data point

 N_{p} = cycles at failure for a particular data point

n = number of test data points

 $\mathbf{N}_{\mathbf{p}}$ = cycles at which data is to be evaluated

Y = test specimen shape constant

 α/\overline{S} = test specimen material constant

The only unknowns are the test specimen shape and material constants. These factors have been evaluated and defined in the prior programs and those values, as shown in Table 1, will be used in this analysis.

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APPENDIX K ENGINEERING DRAWINGS OF SPUR TEST GEARS (CONTINUED)

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APPENDIX M FORMULA FOR DETERMINING EFFECTIVE CASE DEPTH (R/C 50) ON X2M CARBURIZING STEEL

FORMULA FOR DETERMINING EFFECTIVE CASE DEPTH

(R 50) ON X-2 CARBURIZING STEEL

STANLEY E. CASPER CORPORATE METALLURGIST C. I. HAYES INC. 9/23/82

M-1

F.

FORMULA FOR DETERMINING EFFECTIVE CASE DEPTH

(R_50) ON X-2 CARSURIZING STEEL

INTRODUCTION

The carburization of hi-chrome materials requires one to address the problem of the tenacious oxide which serves to block carbon diffusion.

Vacuum carburization allows one to reduce the oxide at a relatively high temperature before cooling to a suitable carburization temperature. The choice of the latter is dependent on the complexity of the part (case uniformity) and the ability to break up the carbides. Generally, the higher temperature one carburizes hi-chrome alloys, the more difficult it is to dissolve the carbides.

After carbunization it is desirable to diffuse at a temperature at least 50° F. higher in order to accelerate carbide solution.

The possibility of interrupting the process to do some intermediate machining always remains, therefore, allowances must be made for the "diffusion" that occurs during the final hardening process.

All of these factors constitute a multi-phase carburize and diffusion technique.

- PHASE 1 Carburize (And diffuse if same temp.)
- PHASE 2 Diffuse should temperature change
- PHASE 3 Diffuse and harden sequence usually followed by a re-heat

TWO PHASE PROCESS FORMULA: (See glossary on page M-11 for definition of symbols.) $d = k \sqrt{t}$ Phase 1 $t_2 = \left(\frac{d}{k_2}\right)^2$ $d_2 = k_2 \sqrt{t_2 + c + t_3}$ Phase 2 $d_2 = \frac{k \sqrt{t}}{\sqrt{t_2}} \left(\sqrt{t_2 + c + t_3} \right)$ THREE FHASE PROCESS $d = k \sqrt{t}$ Phase 1 $t_2 = \left(\frac{d}{k_2}\right)^2$ Phase 2 $d_2 = k_2 \sqrt{t_2 + t_3}$ $t_{4} = \left(\frac{d_{2}}{k_{3}}\right)^{2}$ $d_{3} = k_{3}\sqrt{t_{4} + c + t_{5}}$ Phase 3 $d_{3} = \frac{\frac{1}{\sqrt{t_{2}}} \left(\sqrt{t_{2} + t_{3}}\right)}{\sqrt{t_{4}}} \left[\sqrt{t_{4} + c + t_{5}}\right]$

M-3

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FORMULA - continued

ALTERNATE EXPRESSION - TWO PHASE PROCESS

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M-4

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SUBSTANTIATION:

Assume on X-2

The diffusion constant "k" is 95° of 9310 (data for 9310 previously submitted)

TEMP.	<u>9310</u>	<u>X-2</u>
1650° F.	k=.016	¥=.015
1700° F.	k≖.020	k=.019
1750 ⁰ F.	k=.024	k=.023
1800 ^C F.	k≖.030	k=.0285
1850 ⁰ F.	k=.035	k≂.033
1900 ⁰ F.	k=.040	k=.038
:950° F.	k=.045	k=.043

TEST RUN ON 9/20/82:

Carturize at 1800° F, for 150 minutes Diffuse at 1800° F, for 315 minutes

TOTAL....465 minutes

 $\frac{\text{Phase 1}}{\text{d} = k \sqrt[4]{t} = .0285 \sqrt{\frac{465}{69}} = .0793''$

The parts were gas quenched and annealed to simulate intermediate machining. They were then reheated to 1850° F. for 30 minutes and oil quenched.

$$\frac{rhase 2}{t_2} = \left(\frac{d}{k_2}\right)^2 = \left(\frac{.0793}{.033}\right)^2 = 5.775 \text{ hrs.}$$

$$\frac{d_2}{t_2} = \frac{k_2}{\sqrt{t_2 + c + t_3}} = .033 \sqrt{5.775 + .17 + 5}$$

$$\frac{d_2}{t_2} = .084^{"} \qquad \text{M-5}$$

Actual .086" - .090"



5.

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TEST RUN ON 8/30/82

Carburize at 1800⁰ F. for....150 minutes Diffuse at 1800⁰ F. for.....<u>255 minutes</u> TOTAL...... 405 minutes

Phase 1

$$d = k \sqrt{t} = .0285 \sqrt{\frac{405}{60}} = .074^{\circ\circ}$$

The parts were gas quenched and annealed to simulate intermediate machining. They were then reheated to 1850° for 30 minutes and oil quenched.

Phase 2

Actual .075" - .080"



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TEST RUN ON 8/20/32

Carburize at 1750⁰ F. for.....90 minutes Diffuse at 1750⁰ F. for.....<u>5 minutes</u> TOTAL.....95 minutes

Phase 1

$$d = k \sqrt{t} = .023 \sqrt{\frac{95}{60}} = .029"$$

The parts were gas sugniched and annealed for simulation purposes. They were reheated to 1850 F. for 30 minutes and oil quenched.

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$$\frac{0hase 2}{t_2} = \left(\frac{d}{k_2}\right)^2 = \left(\frac{.029}{.033}\right)^2 = .72 \text{ hours}$$

$$d_2 = k_2 \sqrt{t_2 + c + t_3} = .033 \sqrt{.772 + .17 + .5}$$

$$d_2 = .040^{\prime\prime}$$

Actual parts revealed .048". Since this error was greater than 10% a check was made on furnace temperature. It was discovered that a temperature error of 30° F. existed, whereby the parts actually saw 1780° F. and 1880° F., respectively.

A recalculation substituting

k as .026 k₂ as .036

d₂ = .045"

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TEST_RUN_ON_6/16/82

Carburize at 1750⁰ F. for.....90 minutes Diffuse at 1750⁰ F. for.....<u>4 minutes</u> TOTAL.....94 minutes

•

<u>Phase 1</u>

$$d = k \sqrt{t} = .023 \sqrt{\frac{94}{60}} = .029''$$

The parts were gas quenched and annealed. They were reheated to 1850⁰ F. for 20 minutes and oil quenched.

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$$\frac{Phase 2}{t_2} = \left(\frac{d}{k_2}\right)^2 = \left(\frac{.029}{.033}\right)^2 = .7/2 \text{ hours}$$

$$d_2 = k_2 \sqrt{t_2 + c + t_3} = .033 \sqrt{.772 + .17 + .333}$$

$$d_2 = .0373$$
Actual .038"

TEST RUN ON 6/18/82

SAME CYCLE AS ABOVE

Actual .040"

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M-8

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TEST RUN ON 11/14/79

Carburize at 1650⁰ F. for 30 minutes

Phase 1

$$d = k\sqrt{t} = .015\sqrt{.5} = .0106''$$

Parts were heated to 1850° F. and soaked 40 minutes (this was not a reheat, no constant applied).

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Phase 2

$$t_2 = \left(\frac{d}{k_2}\right)^2 = \left(\frac{.0106}{.033}\right)^2 = 0.10 \text{ hours}$$

$$d_2 = k_2 \gamma t_2 + t_3 = .033 \gamma \cdot 10 + .67 = .029^{\circ\circ}$$

The parts were gas quenched, reheated to 1850° F. for 40 minutes and oil quenched.

$$\frac{Phase 3}{t_4} = \left(\frac{d_2}{k_3}\right)^2 = \left(\frac{.029^n}{.033}\right)^2 = .77 \text{ hours}$$

$$d_3 = k_3 \sqrt{t_4 + c + t_5} = .033 \sqrt{.77 + .17 + .67} = .042^n$$

*Actual .036"

*NOTE: This is not within the <u>+</u> 10%. Impossible to determine if furnace problem existed because of date of test. Sample examination indicated a lean case indicating some problem (most likely temperature) as in 8/20/82 test.

This will be rechecked when time allows.



TEST RUN ON 6/22/79

9.

Carburize at 1650° F. for 17 minutes

Phase 1

 $d = k \sqrt{t} = .015 \sqrt{.283} = .008''$

Parts were heated to 1950⁰ F. for 40 minutes. (No reheat utilized, therefore no constant was used).

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$$\frac{Phase 2}{t_2} = \left(\frac{d}{k_2}\right)^2 = \left(\frac{.008''}{.043}\right)^2 = .035 \text{ hours}$$

$$\frac{d_2}{t_2} = \frac{k_2}{\sqrt{t_2 + t_3}} = .043 \sqrt{.035 + .67} = .036$$

Actual .040"

TEST FUN ON 5/30/79

Same as 6/22/79

M-10

GLOSSARY OF SYMBOLS

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- Initial effective case depth in thousandths of an inch formed in first carburize and/or diffuse at any single temperature
- k Diffusion constant for temperature used in PHASE 1
- Time in hours for the first carburize and/or diffusion at any single temperature
- t2 Time in hours required to create "d" at new PHASE 2 diffusion temperature
- $k_2^{\rm c}$ = Diffusion constant for temperature used in PHASE 2.
- d_2 Effective case depth in thousandths of an inch formed after diffusion in PHASE 2
- t_3 Time of diffusion in hours in PHASE 2
- Reheat constant in hours, i.e., one inch cross section heated to 1800° F. - 1950° F. is .17
- t_4 Time in hours required to create d_2 at a new PmASE 3 temperature
- $k_{\rm B}$ Diffusion constant for temperature used in PHASE 3.
- d_{3} Effective case depth in thousandths of an inch formed after diffusion in PHASE 3 $% \left({\left({{{{\rm{T}}_{\rm{s}}}} \right)} \right)$
- t_5 Time of diffusion in hours in PHASE 3

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APPENDIX N HAYES VACUUM CARBURIZING EVALUATIONS FOR 9310

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**. Roy Cunningham September 22, 1982 Fage 5

AISI 9310 HARDNESS SURVEY

.002R _c 66.5
.005R _c 65
.010R _c 63.5
.015R _c 63.5
.020R _c 63.5
.025R _c 65
.030R_62.5
.040R _c 63.5
.250R _c €3.5
.060R _c 62.5
.070R _c 58
.080R _c 54
.085R _c 50.5
.090R _c 48

As before Roy, I would appreciate your carbon data to correlate with our Leco.

Thank you.

Very truly yours,

C. I. HAYES INC.

· Carp 1.C

Stanley E. Casper Corporate Metallurgist

SEC:esc

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Mr. Roy Cunningham September 22, 1982 Page 4

On August 26, 1982, a test run was made on AISI 9310 to confirm the charts already submitted to Boeing.

The object of this test was to produce .085" effective case depth.

The cycle was:

Heat to 1900⁰ F.

Soak at 1900° F. 30 minutes

Carb at 1900⁰ F. 60 minutes

CH_A = 250 Torr = 7ECCF/rr.

Diff. at 1900⁰ F. 210 minutes

Gas Quench

Reheat to 1550⁰ F.

Soak at 1550° F. 40 minutes

0il quench

Freeze at -120° F. 120 minutes

N-2



APPENDIX O GEARED ROLLER TEST MACHINE DRAWINGS

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APPENDIX O GEARED ROLLER TEST MACHINE DRAWINGS (CONTINUED)

0-2



Heat Treat	Specimen I.D.	Start Date	Ozi Temp	Roll RPM	Loed (Ibs)	Hertz stress (ksi)	Hrs to Failure	Cycles to Failure	Comments
Summit	14	1/26/83	200*F	1,000	2,605	450	14.3	858,000	
heat treat	1 16	1/27/93	200*6	2 000	2,605	450	30.7	1 642 000	
at ' 900"F	is	1/27/83	200*F	920	2 605	450	291	1,606,320	
	13	2/6/83	200*F	920	2,605	450	173	954,960	i
				TEST	TOPPED				
	15	1/25/83	\$00.k	1,000	2.605	450	21.1	726,000	
Hayes	9	1/25/83	200*F	920	2,605	450	12.5	690,000	
neactreat	3	1/25/83	200*5	920	2.605	450	33	- +-	Oil jet clog -
at 1,900**	12	2/6/83	200'F	1,000	2.605	450	28 0	680,000	invalió data
	13	2/6/83	200*	920	2,605	450	191	1,054,320	l
				TESTS	TOPPED				
	2	1/20/83	200*F	920	2.605	450	16.2	894,240	
Summit	3	1/20/83	200°F	1,000	2,605	450	16 9	1,014,000	
heat treat	1	1/21/83	200*F	920	2,605	450	17 1	943,920	
at 1.900°F	4	1/21/83	200*5	1,000	2,605	450	18 3	1 098,000	Contrad
		1/21/83	200**	920	2,605	450	23.4	1,291,680	Crecked
	2	1/24/83	200%	920	2,605	450	129	712.080	invalid data
				TEST	TOPPED				
		1.24/92	20015	1.000	7.605	150		776 000	
Haves	12	1/26/83	200-6	920	2,605	450	22.05	1 2 2 160	
heat treat	14	1/26/83	200*F	920	2 605	450	65	358 800	
A1 : 60.0%E	7	1/31/83	200'F	920	2,605	450	98	540 960	
0 (300)	10	1/31/83	200**	920	2,605	450	175	966,000	
	13	2/2/83	200*F	920	2.605	450	22 8	1.258,560	
				TESTS	TOPPED				
	1	9/22/83	200°F	920	2,605	450	18 9	1,043,280	
Տստուլ	2	9/26/83	200°F	920	2.605	450	14 2	783,840	
heat treat	3	9/27/83	200*F	920	2,605	450	29 0	1,600,800	
at 1 800°F	4	9/30/83	200°F	920	2,605	450	22 9	1,264,080	
	5	10/3/83	200°F	920	2,605	450	29.4	1.622.880	
	6	10/5/83	200°F	920	2,605	450	114	629,200	
		10///83	200.6	920	2,605	450	22.8	1,258,580	Boller
	8	10/11/83	200%	920	2,605	450	4/6	2,027,520	dupped
	0	10/19/83	2001	920	2,605	450	160 0	3.345,720 - ←	inval-d data
	لمشما			TEST	TOPPED				

APPENDIX P GEARED ROLLER RAW TEST DATA FOR AISI 9310

P-1

	HE	SINGLE TC Vasco At treater	00TH FATIGU 0 X2m Ray [1 "a" (Summ	JE TEST Data Nit gear)			
			SERIAL	HIUUI	LOAD	ING	
	STRESS	CYCLES	NUMBER	NUMBER	STEADY	AL TERNATING	
VASCO SK29572-1							
(MILL HEAT:	274,114	27,000	A100	2	5100	2000	
VASCO)	165,554	37,000	4 700	- ~	0016	3000	
	154,698	56,000		m 4	2900	2800	
	208,978	22,000	003A	-	3900	3800	<u> </u>
	198,122	35,000		2	3700	3600	
	18/,266	53,000			0055 0100	3000	
	241,546	32,000	004A	2	4500	4400	
	230,690	26,000		r) •3	4300	4200	
	1004013			,	221	2002	
VASED	214.406	20,000	0014	-	4000	3900	
SK29572-2	203,550	32,000	;	• ~2	3800	3700	
	192,694	39,000		~ 1	3600	3500	
RESID CARTECH)	170 082	2000 000	0020	4 ~	3200	3300	
	160,126	228,000		5	3000	2900	
	142,270	8,215,000		~ ~	2800	2700*	
	150,126	000 956 11	003A	* *	3000	2000	
	181,838	95,000		~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	3400	3300	
	170,982	5,563,000		~ ·	3200	3100	
	18/,266 208 078	000.6/6.6	0040	4~	3000	3400	
	203,550	0000.6		- 2	3800	3700	_
	198,122	19,000		ŝ	3700	3600	
	192,694	20,000		4	3600	3500	
*RUNDUT, NO FAILURE							

APPENDIX Q SINGLE TOOTH BENDING FATIGUE TEST RAW DATA

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(NG	AL TERNATING	4000 4200*	200000 330000 34000000000000000000000000	3900 3900 3700	3400* 3800 3200* 3400 3400	3400* 3400* 4500 4500 4500 4500 4500 4500 4000 40	3800
LOAD	STEADY	4000 4300	4 / 00 4 4 6 00 4 000 6 00 6 00 6 00 6 00 6	4500 4500 3800 3800	3500 3300 3700 3500	3500 3500 4500 4500 4500 4500 4500 4100 4100 4	0065
	TOOTH NUMBER	- 25	י ש – ט י	n 4 – 0 m	8-NW4	ーころはーごうみーごうみーで、	*
	SERIAL NUMBER	001B	002B	0038	0048	0018 0028 0038 0048	
	CYCLES	8,349,000 6,434,000	97,000 9,000 4,758,000 3,037,000	10,000 21,000 13,000 7,000	6,000,000 23,000 8,351,000 39,000 173,000	8,606,000 6,890,000 29,000 379,000 67,000 127,000 127,000 127,000 127,000 127,000 127,000 127,000 127,000 127,000 177,000 25,000 25,000 25,000 23,000 23,000	000.62
	STRESS	214,406 241,546	252,402 246,974 214,406 214,406	241,540 241,546 214,406 214,406 213,550	187,266 208,978 176,410 198,122 187,266	187,256 214,406 241,546 241,546 241,546 246,974 246,974 252,402 252,402 252,402 230,546 219,834 203,550 214,406 214,406	2/18,9/8
		VASCO SK29572-1	(MILL HEAT: 5842A, TELEDYNE VASCO)			VASCO SK29572-2 (MILL HEAT: 86510, CARTECH)	*RUNOUT, NO FAILURE

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SINGLE TOOTH FAITGUE TEST VASCO X211 RAM GATA HEAT TREATER "B" (C. I. HAYES)

Q-2

SINGLE TOOTY BENDING FATIGUE TEST 9310 RAW DATA

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	HEAT TREATER A	(SUMMIT GEAR)	SERIAL	TOOTH	LOADI	ING LB
	STRESS	CYCLES	NUMBER	NUMBER	STFADY	ALTERMATING
	230.690	6,616,000*	001.0	F	4.300	4,200*
SK 29572-10	252,402.	91,000		~	4,700	4,600
(MILL HEAT 87385-2	246.974	8,645,000*		~	4.600	4,500*
CARTECH)	257,830.	6,520,000*		4	4,800	4.700*
	852,635	1,213,000	C03A	-	4,900	4,800
	274,114.	617,000		2	5,100	5.000
	284.970.	190,000		m	0 0E'S	5,200
	290,398.	28.000		-	5,400	5,300
	284,970	2,255.000	0044	-	5.300	5.200
	290,398.	58.000		2	5,400	5,300
	279,542.	4,215,000		m	5,200	5,100
	295,326	1,037.000		4	5.500	5.400

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-RUNOUT, NO FAILURE

Q-3

SPEC	IMEN	TEST	DATES	APPLIED LOAD LB			
NO	TOOTH	START	STOP	STEADY & ALTERNATING	CVCLES #106	REMARKS	APPLIED STRESS
	NO. 1	11-26-85	11-26-85	4,700 ± 4,600	0.084	EAILLIDE	319 676
0014	NO. 2	11-27-85	11-27-85	4,000 ± 3,900	0 022		8/0/077
	NO. 3	12-4-85	12-4-85	4.700 ± 4.600	0520		107'86
	NO. A	12.6.85	13.0.05	2 000 0 0 000		Junour L	2/0'077
		2	CO.C.71	3.80U ± 3./U0	5.327	FAILURE	184,416
0038	NO. 1	12-10-85	12-13-85	3,500 ± 3,400	6.658	FAILLEE	120 223
	NO.1	11-27-85	11 37 eE				500'601
004A			C8-/ 7-1 1	4,000 I 3,400	0.141	FAILURE	194,251
	F ON	11-27-85	12-3-85	4,000 ± 3,900	S.767	FAILURE	194 251

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Q-4

APPENDIX R SCORE TEST RAW DATA

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ROTATING SCORING TEST

9310 RAW DATA

	HEAT TREATER A	(SUMMIT GEAR)
	\$/N	ĨF
SK 29571-1	10A/9A	389.
(MILL HEAT 87885-2		38 8 .
CARTECH)	5A/4A	408.
		409.
	7A/2A	444.
		435.
	3A/17A	406.
		439.
	18A/16A	414.
	"	440.
	21A/19A	422.
	"	411.
	15A/13A	420.
		421.
	12 a/11a	424.
		442.
	8A/14A	413.
		420.
	22A/20A	436.
	"	430.

R-1

ROTATING SCORING TEST

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VASCO X2M RAW DATA

	HEAT TRE (SUMMIT	ATER "A" GEAR)	HEAT TR (C.I.	EATER "B" Hayes)
	s/n	TF	s/n	ŤF
SK29571-1 (MILL HEAT: 5842A, TELEDYNE VASCO)	9A/5A 6A/7A 3A/1A 2A/4A 10A/8A	519 513 538 550 506 532 527 552 542	98/88 108/48 68/58 78/38 38/18	470 489 465 466 485 505 461 484 500 483
SK29571-2 (MILL HEAT: 86:10, CARTECH)	8A/9A 5A/6A 1A/7A 3A/4A 2A/10A	531 496 523 531 523 543 523 523 519 538 532	5B/4B 3B/2B 8B/1B " 7B/6B " 10B/9B	459 472 468 441 471 458 459 495 467 500

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APPENDIX S SPIRAL BEVEL INPUT PINION GEAR DRAWING

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BEAR DATA 884**8**6 SAMA 11

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NUMBER OF STREET		
THE ALLEF LOD FLOT	1-1	
24.435	1 30	
NO OF TEETH	14/30	
MANETRAL PITCH	12000	
PITCH DIA	120425	100
MA JOR DIA	40750	100
MINOR DIA	1.00	
PRESSURE ANGLE	17921	
NASE CIRCLE DIA (REF)		<u>}</u>
SHOW AF HERE N DANENSK	2044	
THEY'S THEME IN MAX EFF WITH GA		+
STATISTICS PIN DIA		+
	-	
	100	- <u>+</u>
PLIFI REAL	1/429	<u>-</u>
THE DA		_
MAL DULLE READINCE	600	<u>.</u>
THE REAL OF THE AND A	OLT WOO	03-00
AND THE PARTY OF THE	100	-
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THE MARCE TEETH 1050 MANETRAL PITCH 6 804 77. 30 .220 CIRCLE AR THICKNESS OF ATTA DUST 116-381 DEDENDUM (REF) MAX TOOTH TO TOOTH SPACING ERROR GLEARANCE. 0003 0015 WAY CUMULATIVE SPACING ERROR PIN DIMENSION MEASURING PIN DIA FILLET RADAYS 045-035 TIE DIA MASE CIRCLE DIA MAX INVOLUTE PROFILE ERROR WAX LEAD ERROR IN /IN 80'0 SHAFT ANOLE 15.0 RH BEIRAL / HELIX ANGLE HAND OF SPIRAL PHELIX 30 91 ETCH_ANGLE DAWER DRIVER OR DRIVEN CC.W_ DIRECTION OF ROTATION BACKLASH CONTRIBUTION OF GEAR .003-.000 WITH MASTER IN PLANE OF ROTATION 11405744
 MATL NO. OF MATING SEAR
 (1003700)

 MARKEL ILETH-MATING SEAR (SEC)
 56

 MORMAL CORRECTED ADDENIOUM (RES).230
 1000.5100 (RES).230

 LINO. 5100 OF TOOTH (RES)
 CONCAVE

 SECURE WITH MATTING SEAR
 CONCAVE

 SECURE OF TOOTH (RES)
 CONCAVE

 MORMAL CORRECTION ADDENIOUN (RES).230
 CONCAVE

 SECURE OF TOOTH (RES)
 CONCAVE

 MORMAL CORDUCTION (RES)
 SIO - .014

 MORMAL COCOCCL THICKNESS
 SIO - .314

 MORMAL COCOCCL THICKNESS
 SIO - .314
 PART NO OF MATING GEAR

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ALLET PADIUS 1 4784 7435.2-•1

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CON OF TOO THE WILL MET AND AND TELLINER CON OF TOO THE ILLUSTING UP MITTEN SMILL MITTN ILLUSTING UP MITTEN SMILL MATCH IN STER WINN '102 OF TAKE MOINT OF OF WILL MATCH MOSTRY WITH CONTACT FROM THE ANTON TOT WITH LONG SMORT ANT LANTTS I TOT WITH 1003 MONTS 100 MELL WITT 1003 MONTS 100

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APPENDIX S SPIRAL BEVEL INPUT PINION GEAR DRAWING (CONTINUED)







APPENDIX S SPIRAL BEVEL INPUT PINION GEAR DRAWING (CONTINUED)
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APPENDIX T BOEING HELICOPTERS MATERIALS ENGINEERING LABORATORY REPORT 88-164

MATERIALS ENGINEERING LABORATORY REPORT

	MELR NO. 89-164
	DATE February 1, 1990
Subject: CH-47C, Combining Transmission Spiral Bevel Input Pi Gear, P/N 114D5245-10, S/N M5373; Metallurgical Eval Vacuum Carburized Gear	nion uation of
Material: 9310 Steel per BMS 7-249, Type III, Vacuum	Carburized
Reference: Army Contract DAAG46-82-C-0034, "Aircraft Quality Temperature Carburizing."	High
Enclosures: I-II Photographic Documentation of Subject Gear III, IV & V Test Data	Pinion
I. OBJECTIVE	
The objective of this investigation was to metallurgical evaluate the subject gear to determine if it conformed to the metallurgical requirements of the engineering drawing.	ly
II. BACKCROUND	
The subject pinion gear was manufactured by Litton Preci- per the requirements of the engineering drawing with the exce- that the pinion was vacuum carburized rather than conventiona carburized, see Reference. Vacuum carburizing was accomplish Summit Gear Corp in Plymouth, Minnesota. The gear was subseq bench tested in a CH-47C combining transmission at Boeing Hel and then submitted to the Materials Engineering Laboratory fo metallurgical evaluation.	sion Gear ption lly ed by uently icopters, r
111. TEST RESULTS	
The as-received spiral bevel pinion gear, P/N 114D5245-1 M5373, is shown in Figure 1. The location of the input pinion a CH-47C combining transmission is illustrated in Figure 2.	0, S/N gear in
Visual and magnetic particle inspection of the gear did reveal any discrepancies.	not
The test data obtained from the subject pinion gear are in Enclosure III. The residual stress profile, which was obta using X-ray diffraction techniques, is presented in Enclosure Carbon profile data, which was obtained using the scanning el microscope and the associated wavelength dispersive spectrome contained in Enclosure V.	contained ined IV. ectron ter, is
T-1	

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MELR 89-164 Page 2

During this evaluation, the only discrepancy noted was that the effective case depths on the flanks of two bevel gear teeth exceeded the requirements of the engineering drawing by 0.001 and 0.002 inch (see Enclosure III). This discrepancy, however, is within the tolerances permitted by Boeing Specification MS 14.02.

<u>NOTE</u>: Although the subject pinion gear was vacuum carburized, rather than conventionally carburized, the carburized areas were evaluated per the requirements of D210-12023-1, i.e., the specification controlling the conventionally carburized pinion gear. In all areas evaluated, the subject pinion gear conformed to the metallurgical requirements of D210-12023-1. For example: The case displayed light discontinuous carbides (Class A-acceptable), there was no evidence of decarburization, there was no visible retained austenite, etc. A typical case and core microstructure is shown in Figures 3 and 4.

IV. CONCLUSION

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In the areas evaluated, the subject CH-47C Spiral Bevel Pinion Gear, P/N 114D5245-10, S/N M5373, conformed to the requirements of the engineering drawing and related specifications.

Prepared by arwood Reviewed by J. Kachelries -Approved by ghan

T-2





T-3









FIGURE 4 TYPICAL CORE MICROSTRUCTURE. STRUCTURE IS PRIMARILY TEMPERED MARTENSITE.

											ENC	ENCLOSURE III	
Boeing He	elicopter	6									HEL	R 89-10	54
114D5245-1	0		M5.			SP	RAL BEVEL	PINION	GEAR				<u></u>
MAGNAGLO No ladicati		·		N : 7 4	LETCH					TH 45			
TETAINED AUST	ENITE			CAR	10 1 HO	WORK .			<u> </u>	111 113			
None Observ	ed				Class /	A			Decart	uriza	tionN	one Obs	served
_	r	_	TYP	A ALLI	MIDE	TYP	E . ALUNINA	77	E C. SIL	CATE	TV	PE 0.03	100
INCLUSIONS	RESUL	<u>T 5</u>	AI-C	AH-	-0	<u> BI-0</u>	<u></u>			-0		5 0H-0	0 NH 1 7
(ASTN 5-45)	RE CHI'S	S .		L.SMAX.	AH-0	<u>i BI-1</u>	Smax BH-0		L. SHAX	CH-U			<u>///-1.5</u>
	CARBON IN	ANG	ANESE	P 100 \$P 1000	<u>Un E</u>	<u>HIGAL</u> NPHUR	BILICON	NICKE	CHR				PPER
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REOM'TE 0	.07-0.13	1.4	-0.7	0.010	Nax 0.0	110max	0.20-0.35	3.00-3.	50 1.0)-1.40	0.08-0	.180.	35ma x
				EFFECTI	VE CASE	DEPTH	AND SURFACE	HARDH	<u>µ</u>				
	GEAR AN	<u>) † E</u>	CTH 1	LINES					CARING	SURPAC			
			CO.	Rc \$9	<u> </u>	PAGE			FECTIVE	CASE D	(C P T H	SURA	ACE
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	T		0.050		64	<u>.</u>		+		10.03	-0.0.0		
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	ROOT 3 FILLETS 4		0.039)	64			0.0	30	0.0	14	64	
0°			0.044	l	64					1			1
·	ROOT	-	0.04		64		0-						
	- FLANKS	1	0.046	5	64			l		1		ĺ	
looth loe			0.04/	<u></u>	- 64		2.7574	0.0	12	0.0	47	64	
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	ROOT	Ť	D.04	3	64		180°	1		1		1	
	+	[0.05	2	63			-		1			
Tooth Heel	PL ANKS	1	0.04	3	64		3.1523			<u> </u>			
0°	ROOT	1	0.040)	64			0.0	33	0.0	48	64	
	PILLETS		0.03	2	64					1		<u> </u>	
	R001	-	0.04	<u> </u>	64			_ <u></u>		∔			
Tooth Heel	FLANKS	;	0.04	s 4	64 64		3.1523			ļ		 	
180*	ROOT	-	0.03	5	63		inch dia	0.0	30	0.0	48	63	
	FILLETS	•	0.03	2	62		1 180*			1		1	
	8001		0.040	0	63			1		1			

		ROOT FILLET DINE	NSIONS	
	DALVE SIDE	NEO'D	HON-DRIVE SIDE	REQ'D
0°	0.045 1nch	0.045-0.055 1nch	0.045 1nch	0,045-0.055 inch

<u>c</u>	DRE HARDNESS O	F BEARING SUR	FACES
Location		Core Actual	Hardness (R/C) Required
2.7574 in d	la 0°	41	32-42
	180°	40	32-42
3.1523 in d	ia 0*	40	32-42
	180*	40	32-42



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CARBON CONTENT IN PINION GEAR TEETH AS A FUNCTION OF DEPTH BELOW THE TOOTH SURFACE.

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