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Interim Report

Heat and Momentum Transfer to Internal
Turbulent Flow of Helium-Argon Mixtures
in Circular Tubes

by Paul E. Pickett

CONVECTIVE HEAT TRANSFER FOR SHIP PROPULSION

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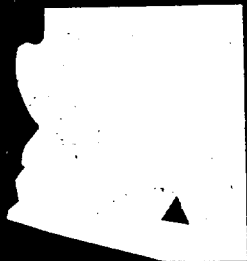
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Interim Report

HEAT AND MOMENTUM TRANSFER TO
INTERNAL, TURBULENT FLOW OF HELIUM-ARGON
MIXTURES IN CIRCULAR TUBES

by

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ABSTRACT

The results of an experimental investigation of friction and heat transfer parameters for turbulent flow of helium-argon mixtures in smooth, electrically heated, circular tubes are presented. Experimental results are compared to existing experimental correlations and to analytical results. Results of air and helium from the same experimental apparatus are included for comparison.

In this experiment helium-argon mixtures with molecular weights between 15.3 and 29.7 are used, ^{He-A} this range resulted in Prandtl numbers between 0.42 and 0.49. Inlet Reynolds numbers range from 31200 to 102000, maximum wall temperatures from 392 to 828°K, maximum wall-to-bulk temperature ratios to 1.82, maximum wall heat flux values to 511 KW/m², and pressures from 469 to 967KPa (4.7 to 9.7 atmospheres).

Existing experimental correlations, developed using gases with Prandtl numbers of approximately 0.7, are compared to the measured friction and heat transfer results. Adiabatic friction factors and friction factors with heat addition are predicted within ±4 and ±10 percent, respectively. Nusselt numbers for fully developed, constant property conditions are predicted within ±5.0 percent. An empirical equation that correlates the helium-argon data within ±15 percent, and includes entrance and variable

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property effects is presented.

Using a recently developed technique that compares numerically calculated and measured constant property Nusselt numbers, turbulent Prandtl numbers in the wall region for helium-argon mixtures are determined. The validity of using these turbulent Prandtl numbers in a variable property numerical analysis is examined. The variation of turbulent Prandtl number with respect to Reynolds number and molecular Prandtl number is also inspected.

TABLE OF CONTENTS

	Page
ABSTRACT	i
LIST OF FIGURES	iv
LIST OF TABLES	v
NOMENCLATURE	vi
INTRODUCTION	1
GAS PROPERTIES	8
EXPERIMENTAL APPARATUS AND PROCEDURE	12
EXPERIMENTAL RESULTS	16
Friction Results	16
Heat Transfer Results	17
NUMERICAL ANALYSIS	28
Procedure for Determining the Turbulent Prandtl Number	28
Procedure for Studying High Heating Rates	33
Turbulent Prandtl Number Results and Discussion	34
High Heating Rate Results and Discussion	36
CONCLUSIONS	39
APPENDICES	42
A. GAS PROPERTIES	43
B. EXPERIMENT	53
Apparatus	53
Procedure	57
Heat Loss Calibration	58
Measurement of Test Section Resistance	64
Meriam Laminar Flow Element Calibration	65
C. UNCERTAINTY ANALYSIS	68
D. HELIUM-ARGON EXPERIMENTAL DATA	71
E. THERMOCOUPLE CONDUCTION ERROR	98
F. RADIATING THERMOCOUPLE CONDUCTION ERROR	105
REFERENCES	112

LIST OF FIGURES

Figure	Page
1. Relative heat transfer coefficients of different gases	2
2. Variation of Prandtl number vs. molecular weight	4
3. Adiabatic friction factors vs. Reynolds number	16
4. Average friction factors vs. modified wall Reynolds number	18
5. Technique for determining constant property Nusselt numbers	20
6. Constant property Nusselt numbers vs. Prandtl number	23
7. Local bulk Nusselt numbers vs. axial position	27
8. Comparison of measured and calculated constant property Nusselt numbers at different turbulent Prandtl numbers	32
9. Calculated and measured bulk Nusselt numbers at different heating rates	37
B1. Schematic diagram of experiment	54
B2. Diagram of test section	55
B3,B4,B5,B6. Heat loss calibration	60-63
B7. Test section resistance vs. temperature	66
E1. Approximation to thermocouple conduction error	100
E2. Predicted thermocouple conduction error for thermocouple 10	103
F1. Radiating thermocouple conduction error for parallel type junction	111

LIST OF TABLES

Table	Page
1. Range of experimental variables	15
2. Variation of turbulent Prandtl number with respect to molecular Prandtl number	34
C1. Uncertainties of measured values	69
C2. Percentage uncertainties in the measured bulk Nusselt numbers	70
D1. Helium-argon adiabatic friction factor data	72
D2. Helium-argon heated flow experimental data	74

NOMENCLATURE

a,	exponent used to account for temperature variation of viscosity;
a_i ,	array of system parameters;
A',	calibration constant for the laminar flow element;
A_{CS} ,	cross sectional area of tube;
b,	exponent used to account for temperature variation of conductivity;
B',	calibration constant for the laminar flow element;
c,	velocity of sound;
c_p ,	specific heat at constant pressure;
D,	inside diameter;
E,	voltage drop;
g,	gravitational constant;
g_c ,	dimensional conversion factor;
G,	mass flow rate per unit area;
h,	heat transfer coefficient;
i,	enthalpy per unit mass;
k,	force constant in Lennard-Jones (6-12) potential;
K,	thermal conductivity;
L,	length between pressure taps in laminar flow element;
l ,	mixing length;
\dot{m} ,	mass flow rate;
\hat{M} ,	molal mass;

ΔP , pressure drop;
 P , power;
 q , heat transfer rate;
 q' , heat transfer rate per unit length;
 q'' , heat flux;
 Q , volume flow rate;
 r , radius;
 R , gas constant for a particular gas;
 R' , resistance per unit length;
 R , universal gas constant;
 T , temperature;
 u , velocity in axial direction;
 x , axial distance from start of heating;
 y , radial distance from wall;
 Y_i , array of measured values;
 Z , a calculated quantity.

Greek symbols

α , thermal diffusivity, $K/c_p \rho$;
 ϵ , force constant in Lennard-Jones potential;
 ϵ_H , eddy diffusivity for heat;
 ϵ_M , eddy diffusivity for momentum;
 γ , ratio of specific heats, c_p/c_v ;
 κ , von Karman constant
 0.4;
 μ , absolute viscosity;
 ν , kinematic viscosity;

ρ , density;
 σ , variance or standard deviation;
 τ , shear stress.

Non-dimensional parameters

f , friction factor, $2g_c \rho \tau_w / G^2$;
 Gr , Grashof number based on wall heat flux,
 $gD^4 q_w'' / (\nu^2 \mu c_p T)_i$;
 M Mach number, j/c
 Nu , Nusselt number, hD/K ;
 Pr , Prandtl number, $c_p \mu / K$;
 q^+ , heat flux parameter, $q_w'' / (Gc_{p,i} T_i)$;
 Re , Reynolds number, GD/μ ;
 y^+ , wall distance parameter, $y(g_c \tau_w \rho)^{1/2} / \nu$;
 y_ℓ^+ empirical constant in van Driest mixing length
 model, 26.

Subscripts

b , evaluated at bulk temperature;
 $cond$, heat conduction;
 cp , constant property condition;
 DB , Dittus-Boelter;
 gen , heat generation;
 i , inlet; an index;
 Max , maximum;
 ref , reference;
 t , turbulent;
 VD , van Driest;
 w , wall;
 Xe , xenon;
 ∞ , environment conditions.

INTRODUCTION

The closed Brayton cycle using inert gases as working fluids has been considered for use in many current applications. The Navy has investigated its use for undersea and surface ship propulsion. NASA has examined it for future space missions requiring relatively large amounts of electric power (100 - 500 Kw) [1,2,3,4]. Binary mixtures of helium and heavier inert gases, such as argon or xenon, have been considered as possible working fluids in these closed Brayton systems. The increase in density, due to the heavier inert gas, reduces the size of the compressor and turbine. The thermal conductivity of the binary mixture is lower than that of helium, thus causing an increase in the size of the heat exchangers. At an intermediate molecular weight an optimum can be attained.

Fig. 1 illustrates the relative heat transfer of helium-argon and helium-xenon mixtures compared to the pure gases and air. The relative heat transfer coefficients were calculated using the Dittus-Boelter type relation

$$h = 0.021 \text{ Re}^{0.8} \text{ Pr}^{0.4} (K/D) \quad (1)$$

and were normalized with respect to the lowest value. The geometry and mass flux were kept constant. This resulted in a relative heat transfer coefficient of the form

$$h/h_{\text{Xe}} = (c_{p\text{Xe}}/\mu_{\text{Xe}}/c_{p\text{He}}/\mu_{\text{He}})^{0.4} (K/K_{\text{Xe}})^{0.6}. \quad (2)$$

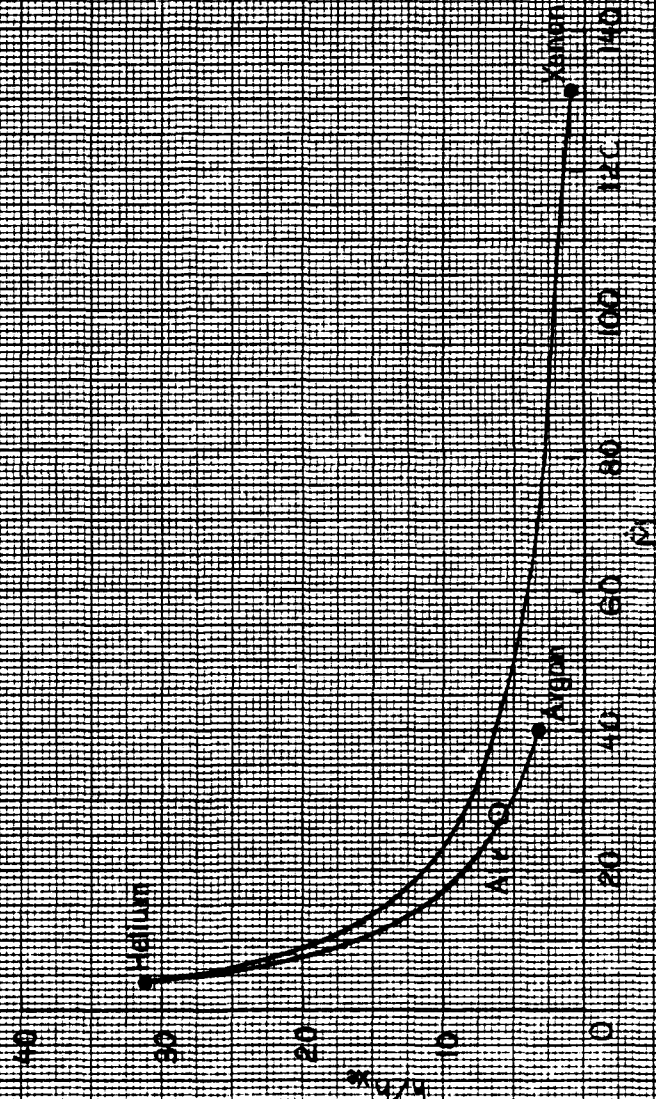


Fig. 1. Relative heat transfer coefficients of mixtures of helium-argon and helium-xenon compared to air and the pure gases, helium, argon, and xenon. Properties were taken at 860K and 101KPa.

All properties were taken at 860K and 101KPa. Vanco [4] performed a similar analysis, but kept the geometry and molal flow rate constant, which gave quite different curves. Examination of Fig. 1 shows why helium-xenon is the prime candidate for a working fluid in the closed Brayton cycle. Helium-argon has been investigated initially due to expense and convenience of experimental apparatus.

The purpose of this research was to determine, for turbulent flow in tubes, the momentum and heat transfer characteristics of helium-argon mixtures. No basic momentum and heat transfer experimental work for fluids with Prandtl numbers between 0.1 and 0.67 presently exists in the literature. Until recently, it was thought that no fluids existed in this Prandtl number range [5,6]. The mixtures of helium and heavier inert gases fill this void, having Prandtl numbers between 0.25 and 0.67. Fig. 2 shows the variation of molecular Prandtl number, Pr , as a function of molecular weight and temperature for helium-argon and helium-xenon [7]. It can be seen that the Prandtl number varies little with temperature.

Experimental correlations, such as equation 1, were developed using air ($Pr \approx 0.7$) and helium ($Pr = 0.67$). Extension of similar experimental correlations for calculating adiabatic friction factors, average friction factors with heat addition, Nusselt numbers at constant property

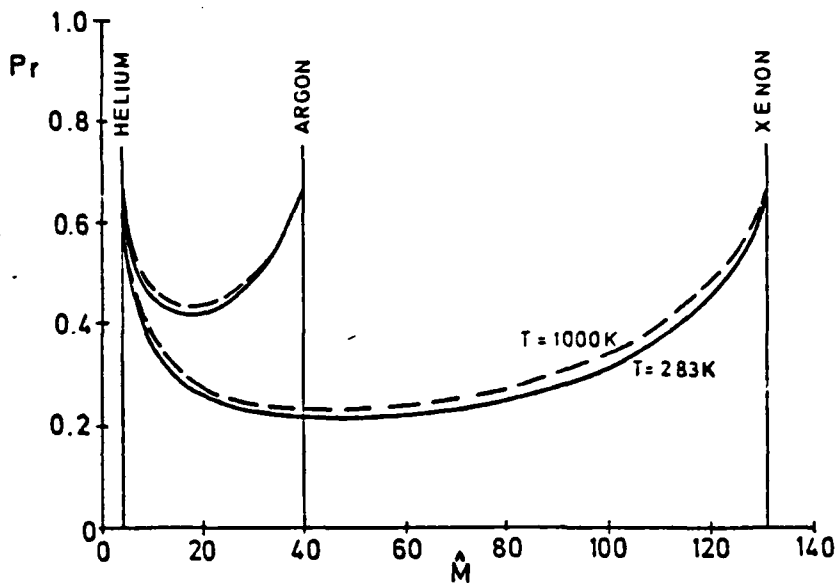


Fig. 2. Variation of Prandtl number with respect to molecular weight for helium-argon and helium-xenon.

conditions, and Nusselt numbers with variable property and thermal entry effects included were examined in this study using helium-argon mixtures. Mixtures at molecular weights of approximately 15 ($Pr = 0.42$), 27 ($Pr = 0.46$), and 30 ($Pr = 0.49$) were used. For comparison, experiments with air and helium were also performed. Experimental studies similar to this one, except using air, helium, or hydrogen include those by Perkins and Worsøe-Schmidt [8], McEligot and Magee [9], Taylor [10], and Dalle Donne and Bowditch [11].

In many analyses that predict turbulent heat transfer results, the value of the turbulent Prandtl number is needed [12]. The turbulent Prandtl number, Pr_t , is defined as the ratio of eddy diffusivity of momentum and eddy diffusivity of heat, ϵ_M/ϵ_H . The eddy diffusivities are defined by the transport relationships,

$$\begin{aligned} \tau/\rho &= (\nu + \epsilon_M) \frac{\partial u}{\partial y} \\ q''/\rho c_p &= -(\alpha + \epsilon_H) \frac{\partial T}{\partial y} \end{aligned} \quad (3)$$

and are used to account for the additional momentum and heat transport caused by turbulent mixing.

Much work has been done, both analytical and experimental, to develop methods to predict Pr_t . As of yet, no generally accepted method exists. Reynolds [13] examined more than thirty ways that have been developed to determine Pr_t . For more background information his review can be consulted. Quarmby and Quirk [14] demonstrated the wide

range of Pr_t values that are predicted by different analyses and measured data. For air and other common gases, they showed that different methods predict Pr_t near the wall from 0.5 to infinity.

Due to large uncertainties [15], experimental measurements haven't clarified the discrepancies. The measurements have indicated that Pr_t is a function of Pr , position in the flow, and turbulence intensity [13]. It has been generally observed that Pr_t increases as the wall is approached, and that the relationship between Pr_t and Pr is [12,13]

$$Pr_t \lesssim 1 \text{ for } Pr \geq 1 \text{ unless } Pr \ll 1. \quad (4)$$

A recent technique, developed by McEligot, Pickett, and Taylor [16], determines Pr_t in the wall region by comparing the experimentally measured and numerically calculated axial variation of Nusselt number. The Nusselt numbers are calculated for the constant properties condition, and the measured Nusselt numbers are extrapolated to a constant properties condition. The technique was used in this investigation to determine Pr_t in the wall region for mixtures of helium-argon. By comparing Pr_t for helium-argon mixtures with results for air [16], the variation of Pr_t as a function of Pr was examined. The variation of Pr_t as a function of Reynolds number was also examined.

Relatively high heating rates could possibly occur

in the heater tubes of the closed Brayton cycle. These high heating rates cause significant variation of properties, and the constant properties idealization becomes invalid. To calculate bulk Nusselt numbers of helium-argon mixtures at these conditions, the Pr_t determined for constant properties was used in a numerical analysis in which the properties were allowed to vary. To validate using Pr_t determined for constant property conditions in a variable properties analysis, calculated and measured bulk Nusselt numbers were compared. By examining this comparison, the possibility that the helium and argon had separated, due to the Soret effect [17], was examined.

GAS PROPERTIES

The properties needed for this study were the compressibility, viscosity, thermal conductivity, specific heat, enthalpy, speed of sound, and gas constant. The properties of air have been studied extensively, and tables listing these properties are readily available. The Tables of the Thermal Properties of Gases [18] were used in this investigation. The properties of helium and helium-argon mixtures were calculated theoretically. For all of the gases, the viscosity and thermal conductivity were assumed to be independent of pressure.

The helium and helium-argon mixtures were assumed to be ideal gases, thus making the compressibility equal to a constant value of one. This is a reasonable assumption for the range of pressures (101.3 - 967.3KPa) and temperatures (294 - 828°K) used in this experiment. Since helium and argon are monatomic, and the temperatures used in this study were not too great, the equation [19]

$$c_p = (5/2) R \quad (5)$$

was used to calculate the specific heat. The specific heat was assumed to be constant, and the gas constant was calculated from the relation

$$R = R/\hat{M}. \quad (6)$$

Using the ideal gas and constant specific heat assumptions, simple equations for the enthalpy and speed of sound can

be derived [20]

$$i = c_p (T - T_{ref}) \quad (7)$$

$$c = \sqrt{\gamma R T} = \sqrt{5/3 R T} \quad (8)$$

T_{ref} is an arbitrary reference temperature. From the assumptions already mentioned, the ratio of specific heats, γ , becomes a constant value of 5/3.

The viscosity and thermal conductivity of the helium and helium-argon mixtures were calculated using the Lennard-Jones (6-12) potential in the Chapman-Enskog kinetic theory [17]. The predicted properties were compared with experimental measurements.

The force constants, ϵ/k and σ , suggested by Hirschfelder, Curtiss and Bird [17] were tried originally. The predicted properties were compared with the experimental values only for the range of temperatures used in this study. The predicted helium viscosities were five percent below the experimental measurements of Dawe and Smith [21] and Kalelkar and Kestin [22]. The predicted thermal conductivities of helium agreed within one percent of the measurements by Saxena and Saxena [23], but were five percent below the values calculated from experimental viscosity measurements of Kalelkar and Kestin [22]. The predicted viscosities of helium-argon mixtures at 870°K were three to five percent below the measured values of Kalelkar and Kestin [22]. The predicted thermal conductivities of helium-argon mixtures at 790°K were five to

nine percent below the measured values of the Thermophysical Properties Research Center [24], and the measured values of von Ubisch repeated by Gandhi and Saxena [25].

In an attempt to get better agreement between predicted and measured values, force constants suggested by DiPippo and Kestin [26] were tried. With these force constants, the predicted viscosities of both helium and helium-argon agreed within one percent of the measured values mentioned in the previous paragraph. The predicted thermal conductivities of helium agreed within one percent of the values of Kalelkar and Kestin [22], but were five percent above the measurements of Saxena and Saxena [23]. The predicted thermal conductivities of helium-argon were essentially unchanged. Since the agreement between the predicted and measured viscosities was improved, and the agreement between the predicted and measured thermal conductivities remained approximately the same, the force constants suggested by DiPippo and Kestin [26] were used. The calculated properties of helium and the helium-argon mixtures used in this investigation are listed in Appendix A.

The properties were inserted in tabular form in the numerical programs that reduced the experimental friction and heat transfer measurements. In the numerical program used to predict heat transfer results, the properties were inserted in equation form. The ideal gas law was used, the specific heat was assumed constant, and the variation

of viscosity and thermal conductivity with temperature was accounted for with the following relations.

$$\mu/\mu_{\text{ref}} = (T/T_{\text{ref}})^a \quad (9)$$

$$K/K_{\text{ref}} = (T/T_{\text{ref}})^b \quad (10)$$

As discussed by McEligot, Taylor, and Durst [7], the exponent "a" ranges from 0.7 to 0.8, and the exponent "b" ranges from 0.7 to 0.75 for the inert gases and their mixtures. The exponents, "a" and "b", of air for the range of temperatures in this study are 0.67 and 0.81, respectively. Thus, the viscosity and thermal conductivity of air, helium, and helium-argon vary with temperature in approximately the same manner.

For the present study the following values of the exponents were used for the mixtures:

$$\text{at } \hat{M} = 15.83, \quad a = 0.745 \text{ and } b = 0.718$$

$$\text{at } \hat{M} = 27.53, \quad a = 0.772 \text{ and } b = 0.741.$$

EXPERIMENTAL APPARATUS AND PROCEDURE

The experimental apparatus, arrangement, and procedure was similar to that used by Perkins, Schade, and McEligot [27]. Only differences in the two experiments will be noted here. Instead of a square duct, a circular tube made of Hastelloy-X was used as a test section. The tube had an inside diameter of 0.312 cm. and a wall thickness of 0.056 cm. The test section consisted of a heated section 98 diameters in length preceded by an unheated section 92 diameters in length. The unheated section ensured that the velocity profile was fully developed at the inlet of the heated section. For attachment of the a.c. power cables, stainless steel electrodes were brazed at the upper and lower ends of the heated section. Two pressure taps were used. One was located in the lower electrode and the other 8.0 diameters below the upper electrode. Sixteen premium grade chromel-alumel thermocouples (0.013 cm. diameter) were spot welded to the heated section of the tube using the parallel junction suggested by Moen [28].

In addition to the power supply used by Perkins et al. [27], an a.c. Lincoln welder was used in order to reach the high temperatures at the larger Reynolds numbers used in this experiment. To measure the higher flow rates, the positive displacement meter was replaced by a Meriam

laminar flow element. The latter was calibrated to measure the flow rate within ± 1.5 percent. Heise gages, inclined water manometers, and vertical mercury or water manometers were used to measure static pressure and pressure drop.

A vacuum external environment was not used in this experiment. The test section was completely enclosed with a heat shield that restricted the convective air currents and helped stabilize the heat loss from the tube to the environment.

The experimental procedure was slightly different than that used by Perkins et al. [27]. The "radiating thermocouple conduction error", discussed by Hess [29], was not exactly appropriate since the test section was surrounded by air at atmospheric pressure. Instead a correlation for natural convection from small wires was introduced, in addition to radiation, as detailed in Appendix E. The heat loss from the tube to the environment was determined using the method described by Campbell and Perkins [30].

To reduce the heat transfer data the same computer program that was used by Perkins et al. [27], was employed in this study, but was modified for use with a circular tube. The basics of this computer program are described

in other reports [30,31,32]. Table 1 summarizes the range of variables covered in this investigation. A more detailed discussion of the experiment is contained in Appendix B. A list of the experimental data is contained Appendix D.

TABLE 1
 Range of Variables in
 the Present Experiment

	Air	Helium	Helium-Argon
Experimental runs	25	4	28
Molecular weight	28.97	4.003	15.30 - 29.70
Inlet bulk Reynolds number	32900 - 100000	30200	31200 - 102000
Exit bulk Reynolds number	19900 - 89000	18400 - 26600	17000 - 68000
Inlet bulk Prandtl number	0.719	0.667	0.419 - 0.486
Exit bulk Prandtl number	0.682 - 0.708	0.667	0.426 - 0.495
Maximum T_w/T_b	1.90	1.75	1.82
Maximum T_w ($^{\circ}K$)	817	789	828
Maximum q^+	0.0027	0.0027	0.0032
Maximum Gr/Re_i^2	8.90×10^{-5}	4.84×10^{-5}	3.22×10^{-3}
Maximum Mach number	0.26	0.25	0.33
x/D for local bulk Nusselt numbers	2.1 - 82.0	2.1 - 82.0	2.1 - 82.0

EXPERIMENTAL RESULTS

Friction Results

Adiabatic friction factors were measured before each series of heated runs. These were compared to other researcher's results, and were also used as a check of the pressure, mixture molecular weight, and flow rate measurements. The method described by Shapiro [33] was used to calculate the adiabatic friction factors. The measured friction factors were compared to the experimental correlation of Drew, Koo, and McAdams [34],

$$f = 0.0014 + 0.125 \text{ Re}^{-0.32}. \quad (11)$$

This correlation is for turbulent flow in tubes, and was used because of its simplicity and close agreement with the Kármán-Nikuradse relation. Fig. 3 shows the measured friction factor divided by that calculated from equation (11) plotted as a function of Reynolds number. Air and helium data points are included for comparison. All the measured friction factors are within ± 4.0 percent of equation (11), and 76 percent are within ± 2.0 percent.

Since only two pressure taps were used, local friction factors could not be determined for experiments with heat addition. Average friction factors were determined in the manner of Humble, Lowdermilk, and Desmon [35]. The average friction factors were compared to an experimental correlation suggested by Taylor [36]. This correlation is for

turbulent flow in tubes with heat addition.

$$f = (0.0014 + 0.125 \text{Re}_w^{-0.32}) (T_w/T_b)^{-0.5} \quad (12)$$

This relation is similar to equation (11), but the bulk Reynolds number is replaced by the modified wall Reynolds number. The term $(T_w/T_b)^{-0.5}$ is included to account for variation of properties with temperature. Equation (12) was used by Taylor to correlate average friction coefficients measured by several different people. It predicted most of the data within ± 10 percent.

Fig. 4 shows the average friction factors with heat addition as measured in this investigation. The friction coefficients are divided by equation (12) and plotted as a function of modified wall Reynolds number. Again, helium and air are included for comparison. All of the data is predicted to within ± 10 percent by equation (12) and 84 percent is predicted to within ± 4.0 percent.

Heat Transfer Results

To determine the effects of the lower helium-argon Prandtl number on the heat transfer results, the variation of properties with temperature, and the entrance effects were minimized. The entrance effects were minimized by considering primarily the results at which fully developed conditions existed ($x/D > 20$). A method described by Malina and Sparrow [37] was used to approach the constant properties idealization.

For the method described by Malina and Sparrow, a

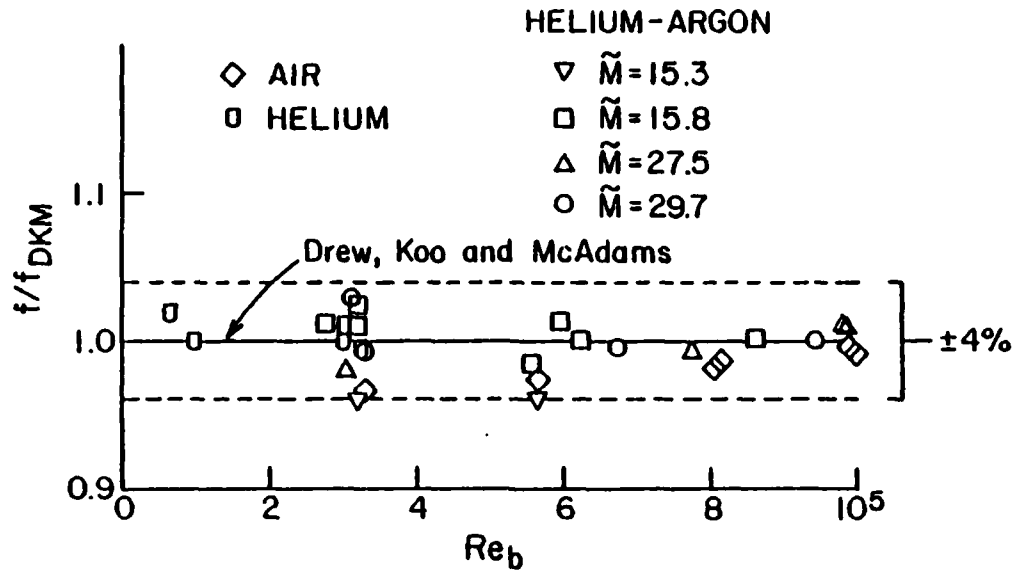


Figure 3. Comparison of Adiabatic Friction Factors to Drew, Koo and McAdams Correlation for Air, Helium and Helium-Argon Mixtures.

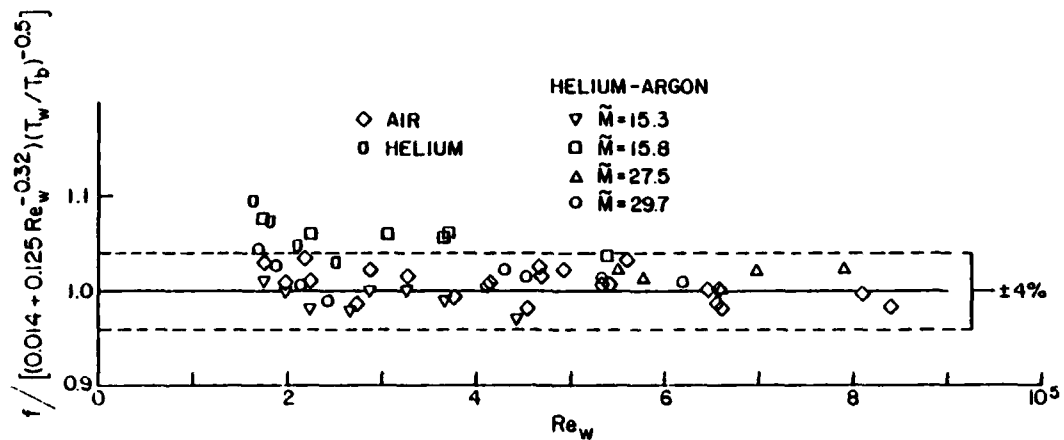


Figure 4. Comparison of Average Friction Factors to Taylor Variable Properties Correlation for Air, Helium and Helium-Argon Mixtures.

fixed inlet Reynolds number is maintained while the wall-to-bulk temperature difference is varied. At a particular axial location, the ratio of experimentally determined bulk Nusselt number to a Dittus-Boelter type correlation is plotted as a function of the difference between wall and bulk temperature. Extrapolation to a difference of zero between the wall and bulk temperature gives a ratio that can be directly used to calculate a constant property Nusselt number, Nu_{cp} . Since the ratio of bulk Nusselt number to a Dittus-Boelter type correlation partially eliminates any effects caused by small deviations of the Reynolds number, these deviations should be kept as small as possible.

The procedure described in the previous paragraph is demonstrated in Fig. 5 for a helium-argon mixture with a molecular weight of 15.30, inlet Reynolds number of 55200, and inlet Prandtl number of 0.419. Extrapolation for four different axial locations is shown. For this investigation the Dittus-Boelter type correlation used was (equation 1 rearranged)

$$Nu_{DB} = 0.021 Re^{0.8} Pr^{0.4}. \quad (13)$$

For a sequence of runs at a nominal inlet Reynolds number, all individual runs had inlet Reynolds numbers within 1.8 percent of the nominal value.

The dashed lines in Fig. 5 show how the error in the constant property Nusselt number was estimated. This

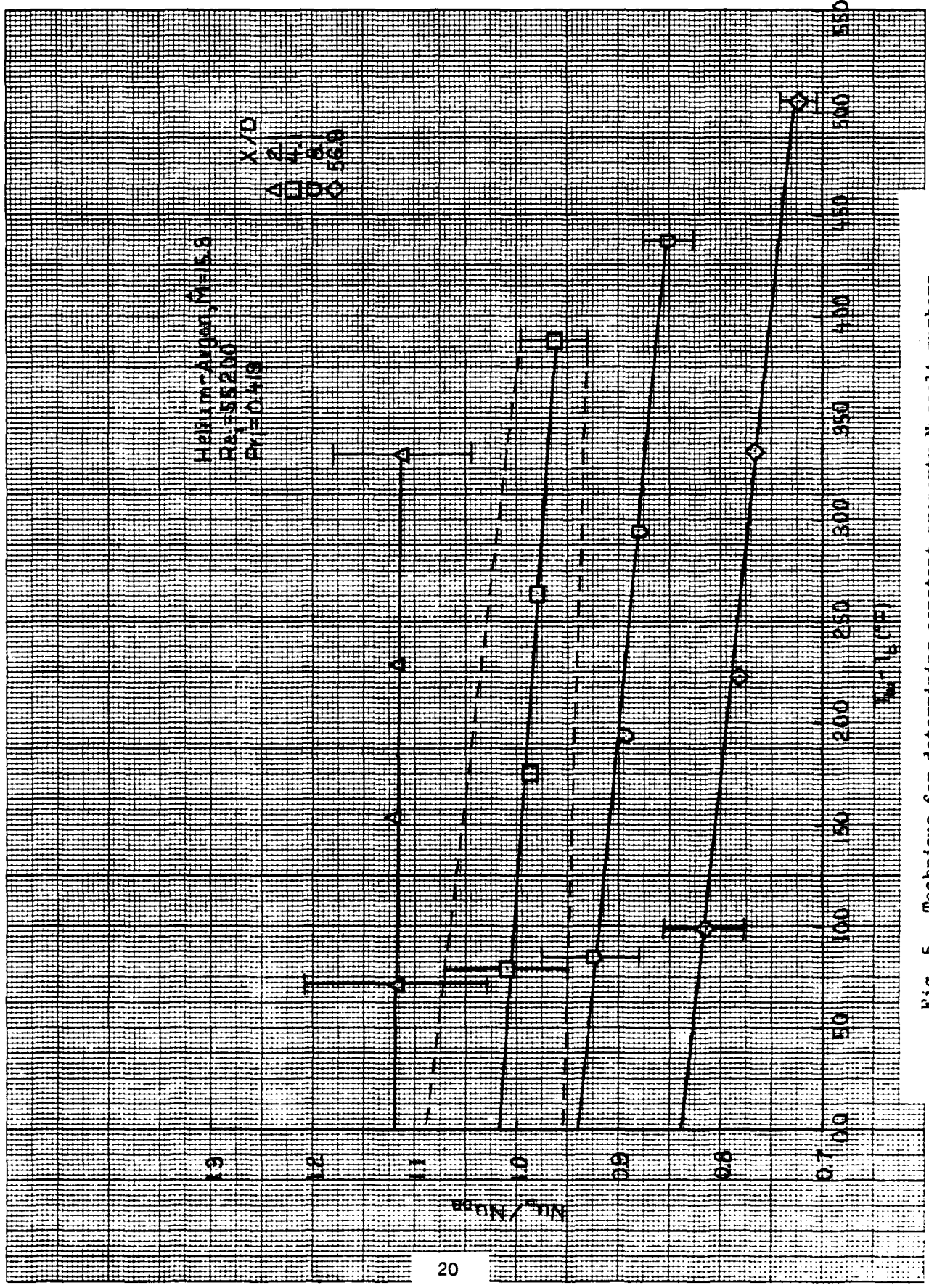


Fig. 5. Technique for determining constant property Nusselt numbers.

technique was used by Reynolds, Swearingen, and McEligot [38]. The error in this investigation varied from ± 9 percent at small x/D to ± 5.4 percent at large x/D . The dominant uncertainty in the Nusselt number is due to uncertainty in the wall-to-bulk temperature difference. This difference is small in the entrance region, thus causing a large error in the Nusselt number. At large x/D , the flow is fully developed and the wall-to-bulk temperature difference is relatively constant. The error in the Nusselt number becomes a minimum, and then increases with increasing x/D due to greater uncertainty in the bulk temperature. Appendix C describes the method used for calculating error in the measured Nusselt number.

For fully developed conditions ($x/D > 20$), with air or helium as the experimental fluid, the ratio Nu_{cp}/Nu_{DB} varied from 0.94 to 1.00. No dependence on Reynolds number was noticed for the Reynolds number range used in this experiment. For fully developed conditions, with helium-argon mixtures as the experimental fluid, the ratio Nu_{cp}/Nu_{DB} varied from 0.83 to 0.93. For a helium-argon mixture at a molecular weight of 15.30, inlet Reynolds number of 55200, and x/D value of 56.9, Fig. 5 shows the ratio Nu_{cp}/Nu_{DB} to be approximately 0.84. From these results, it was determined that the Dittus-Boelter type equation (equation 13) did not predict correct Nusselt numbers for

the Prandtl number range between 0.42 and 0.50.

A correlation suggested by Kays [39] predicted the constant property Nusselt numbers of the helium-argon mixtures within ± 6.0 percent.

$$\text{Nu} = 0.022 \text{Re}^{0.8} \text{Pr}^{0.6} \quad (14)$$

This equation was recommended for fluids with Prandtl numbers between 0.5 and 1.0, constant properties, a constant heat flux boundary condition, and fully developed turbulent flow. If the coefficient of this equation is changed to 0.021, and the Prandtl number exponent adjusted so that approximately equivalent results are obtained, the resulting equation is

$$\text{Nu} = 0.021 \text{Re}^{0.8} \text{Pr}^{0.55} \quad (15)$$

This equation shows that the exponent of the Prandtl number in equation (13) should be changed from 0.4 to 0.55 in order to accurately predict the constant property Nusselt numbers of the helium-argon mixtures. Fig. 6 shows the constant property Nusselt number divided by equation (15) plotted as a function of Prandtl number for the mixtures. Results are plotted for three Prandtl numbers, four Reynolds numbers, and axial positions at which the flow was fully developed. From the figure it can be seen that equation (15) predicts the constant property Nusselt numbers within ± 5.0 percent. At a Reynolds number of 32000, a small effect of the Prandtl number varying between 0.419 and 0.486 can be noticed.

A dependence on Reynolds number was observed for the

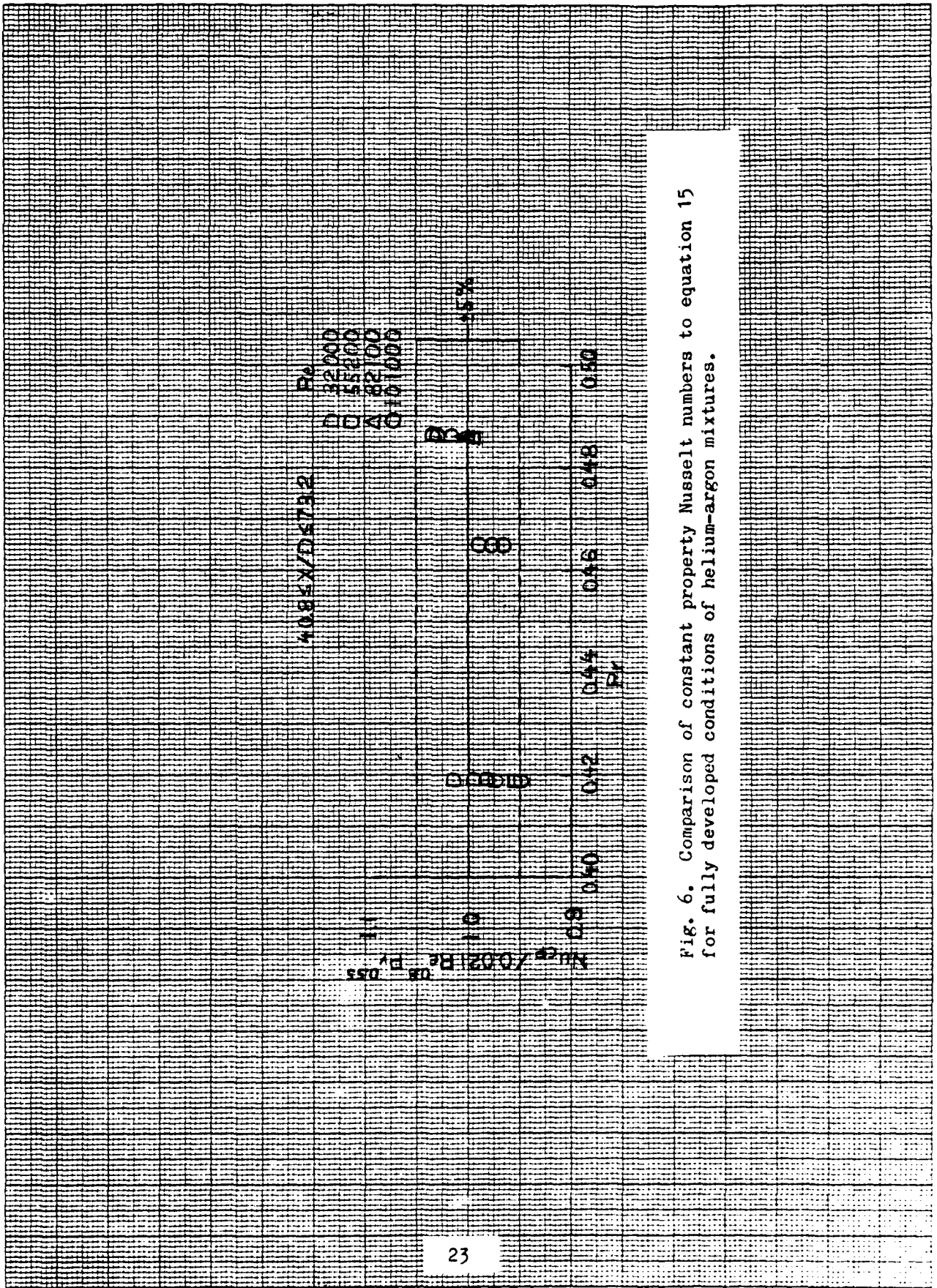


Fig. 6. Comparison of constant property Musselt numbers to equation 15 for fully developed conditions of helium-argon mixtures.

helium-argon mixtures. Since the effect on the constant property Nusselt number was about equivalent to the error in the constant property Nusselt number, only general trends can be discussed. Two trends were observed (Fig. 6). For a particular Prandtl number the ratio of Nu_{cp} divided by equation (15) decreased as the Reynolds number increased, and this effect became more pronounced as the Prandtl number decreased.

To account for the variation of properties and entrance effects in this investigation, the correction factors suggested by Magee [40] were used.

$$[(T_w/T_b)^{-0.4} + 0.6D/x] \quad (16)$$

The term $(T_w/T_b)^{-0.4}$ accounts for the variation of properties, and the term $0.6D/x$ accounts for the entrance effects. If these correction factors are applied to equation (13), the resulting equation is

$$Nu_b = 0.021 Re_b^{0.8} Pr_b^{0.4} [(T_w/T_b)^{-0.4} + 0.6D/x]. \quad (17)$$

For x/D between 2.1 and 81.6 this equation predicted all of the present measured Nusselt numbers for air and helium within ± 15 percent and 97 percent of the Nusselt numbers within ± 10 percent.

If the correction factors (16) are applied to equation (15) the resulting relation is

$$Nu_b = 0.021 Re_b^{0.8} Pr_b^{0.55} [(T_w/T_b)^{-0.4} + 0.6D/x]. \quad (18)$$

This equation predicted the helium-argon Nusselt numbers in the fully developed region within ± 13 percent, but

underpredicted the Nusselt numbers in the entrance region by as much as 22 percent. To have the same type of accuracy with the helium-argon data that was obtained with the air and helium data changes to the correlation were necessary.

As previously discussed, the transport properties of helium-argon vary with temperature in approximately the same manner as those of air and helium. For this reason the term $(T_w/T_b)^{-0.4}$ was retained as a reasonably accurate correction factor for the variation of properties. Kays [39] discusses the effect of different Prandtl numbers in the thermal entrance region of circular tubes. He shows that as the Prandtl number decreases, the effect of the entrance region on the Nusselt number is more pronounced. Because of this, the coefficient in the term $0.6D/x$ of equation (18) was changed. Since helium-argon has a lower Prandtl number than air, one would expect the coefficient to have a larger value than 0.6. Different values for the coefficient of the entrance effects term were used in equation (18), and compared to the experimentally determined bulk Nusselt numbers of helium-argon. From this comparison it was determined that a value of 0.85 worked best for the coefficient of the entrance effects term. The complete correlation, accounting for entrance effects and variation of properties is

$$Nu_b = 0.021 Re_b^{0.8} Pr_b^{0.55} [(T_w/T_b)^{-0.4} + 0.85D/x]. \quad (19)$$

For x/D between 2.1 and 81.6 this equation predicted all

of the measured bulk Nusselt numbers for helium-argon within ± 15 percent and 92 percent within ± 10 percent.

Measured bulk Nusselt numbers divided by equation (19) are plotted on Fig. 7 as a function of x/D . For clarity, only results from four helium-argon experimental runs were plotted. The data plotted are from experimental runs that include the complete range of experimental variables for the helium-argon mixtures. The greatest difference between the experimental data and equation (19) occurred at high heating rates in the x/D range between 4.0 and 16.0. In this range equation (19) underpredicted the measured Nusselt numbers by 5 to 15 percent.

Few correlations for gases with Prandtl numbers between 0.1 and 0.67 presently exist in the literature. Sleicher and Rouse [41] suggest a correlation for Prandtl numbers between 0.1 and 10^5 , and Reynolds numbers between 10^4 and 10^6 . The correlation is for fully developed conditions, and accounts for property variation.

$$\begin{aligned} \text{Nu}_b &= 5 + 0.015 \text{Re}_f^m \text{Pr}_w^n \\ m &= 0.88 - 0.24/(4 + \text{Pr}_w) \\ n &= 1/3 + 0.5 \exp(-0.6 \text{Pr}_w) \end{aligned} \quad (20)$$

For the helium-argon mixtures, this equation predicted Nusselt numbers that were 15 to 40 percent lower than the Nusselt numbers measured in the fully developed region of this investigation. Equation (19) correlated the data more accurately.

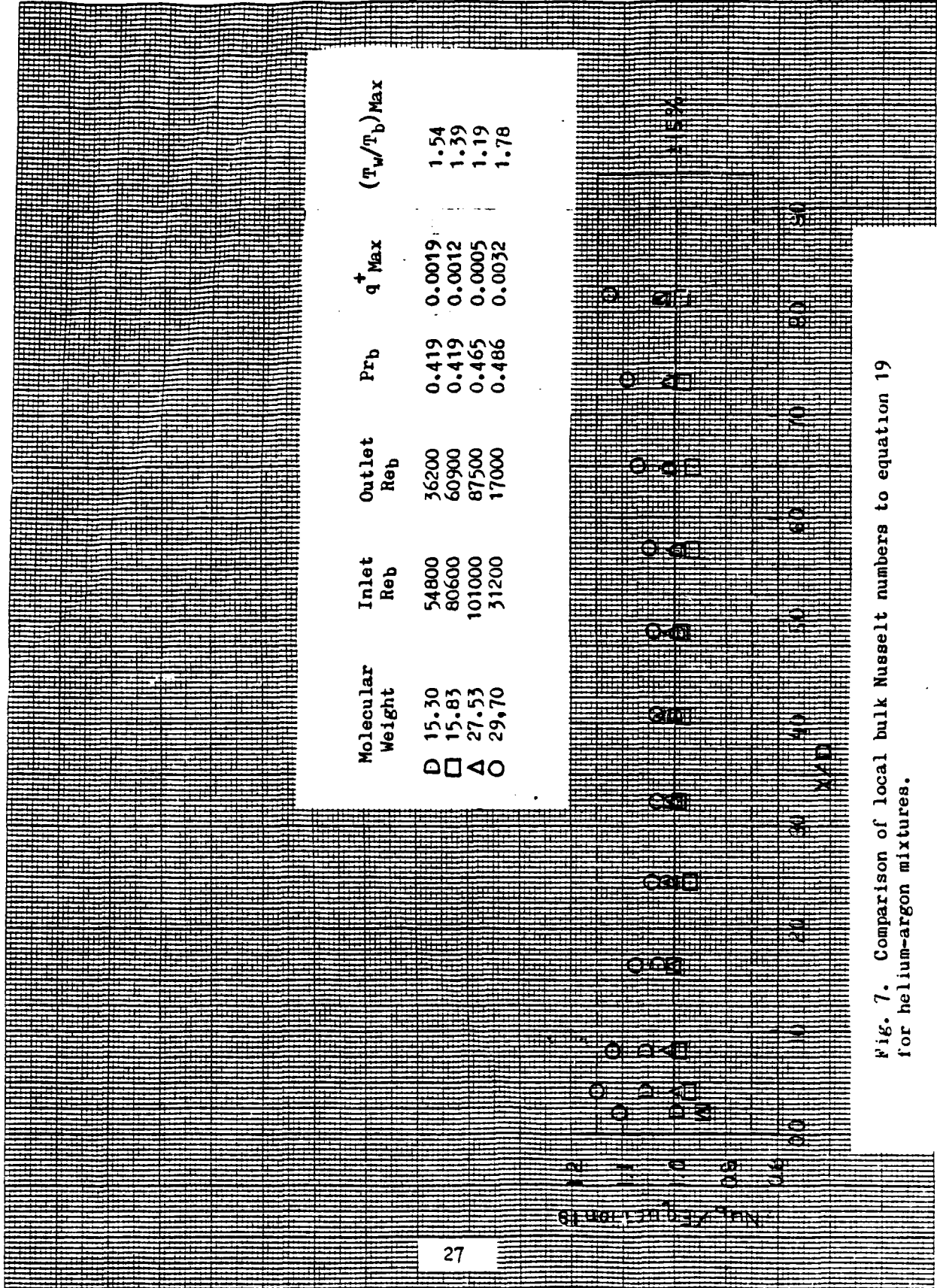


Fig. 7. Comparison of local bulk Nusselt numbers to equation 19 for helium-argon mixtures.

NUMERICAL ANALYSIS

Procedure for Determining the Turbulent Prandtl Number

To determine Pr_t for helium-argon mixtures, the numerical method of Bankston and McEligot [42] was used in conjunction with the technique developed by McEligot, Pickett, and Taylor [16]. The numerical method uses finite control volume approximations. It was developed to solve the coupled, partial differential, axisymmetric, boundary layer equations; but can also be used for constant property conditions which uncouples the boundary layer equations. The boundary conditions are the no-slip and impermeable-wall conditions, the inlet conditions, and the wall heat flux.

The technique of McEligot et al. [16] uses the axial variation of the Nusselt number to determine Pr_t in the wall region. By examination of the simplified energy equation,

$$u \frac{\partial T}{\partial x} = 1/r \frac{\partial}{\partial r} [r(\alpha + \epsilon_M/Pr_t) \frac{\partial T}{\partial r}] \quad (21)$$

they showed that the functional dependence of the Nusselt number is

$$Nu = Nu \{x, u(r), \epsilon_M(r), Pr_t\}. \quad (22)$$

The energy equation was simplified from the general form by using the following assumptions: the axisymmetric boundary layer approximations, hydrodynamic fully developed flow, steady flow at low velocities, and constant fluid

properties. By using one of the semi-empirical relationships for $\epsilon_M(r)$ to determine the velocity profile, $u(r)$, they obtained the result $Nu = Nu \{x, Pr_t\}$. They inverted this relationship to obtain $Pr_t = Pr_t \{Nu(x)\}$. If Pr_t is considered one-dimensional, comparison of experimental measurements of $Nu(x)$ with calculated values of $Nu(x)$ can be used to determine $Pr_t(r)$. McEligot et al. [16] pointed out that direct inversion would be difficult, and iterative use of the numerical procedure described in the previous paragraph was used. The radial variation of the turbulent Prandtl number was assumed to be

$$Pr_r = Pr_{t,w} + \frac{d(Pr_t)}{d(y/r_w)} (y/r_w). \quad (23)$$

The results of McEligot et al. [16] showed that a change of Pr_t in the wall region from 1 to $\frac{1}{2}$ caused changes of 30 to 45 percent in $Nu(x)$, whereas a change of Pr_t in the core only caused small changes. For air at a Reynolds number of 44500 and a Prandtl number of 0.72 they determined that

$$Pr_{t,w} = 0.9 \pm 0.1 \text{ and } \frac{d(Pr_t)}{d(y/r_w)} = 0. \quad (24)$$

The typical errors of the experimentally measured Nusselt numbers did not allow calculation of $\frac{d(Pr_t)}{d(y/r_w)}$.

To determine $\epsilon_M(r)$ in this investigation, the van Driest mixing length model [43] was used in conjunction with the

Reichardt middle law [54].

$$\begin{aligned} \ell_{VD} &= \kappa y [1 - \exp(-y^+ / y_\ell^+)] \\ \epsilon_{VD} &= \ell_{VD}^2 \frac{\partial u}{\partial y} \text{ and } \epsilon_M = \epsilon_{VD} \cdot \left(2 - \frac{y}{r_w}\right) \cdot \left[1 + 2\left(\frac{r}{r_w}\right)^2\right] / 6 \end{aligned} \quad (25)$$

The values of κ and y_ℓ^+ were 0.4 and 26, respectively.

With these constants, the predicted friction factors agreed within one percent of equation (11) for the range of Reynolds numbers used in this study. In this study, as in the study by McEligot et al. [16], the errors in the experimentally measured Nusselt numbers did not allow calculation of $\frac{d(\text{Pr}_t)}{d(y/r_w)}$. The inlet Reynolds number, inlet Prandtl number, constant properties condition, wall heat flux variation, and different values of Pr_t were used as input to the numerical procedure. For the first three diameters, the experimental axial wall heat flux variation resembled an exponential approach to a constant value as x increased. For the remaining length, the wall heat flux was constant within two percent. The same axial variation of wall heat flux was used for all of the constant property numerical calculations.

From the numerical analysis the axial variation of Nu_{cp} was calculated. By comparing graphs of the experimentally measured Nu_{cp} and the calculated Nu_{cp} (examples in Fig. 8), $\text{Pr}_{t,w}$ for helium-argon mixtures was determined. The variation of $\text{Pr}_{t,w}$ with respect to Reynolds number was examined by comparing Nu_{cp} at different Reynolds numbers,

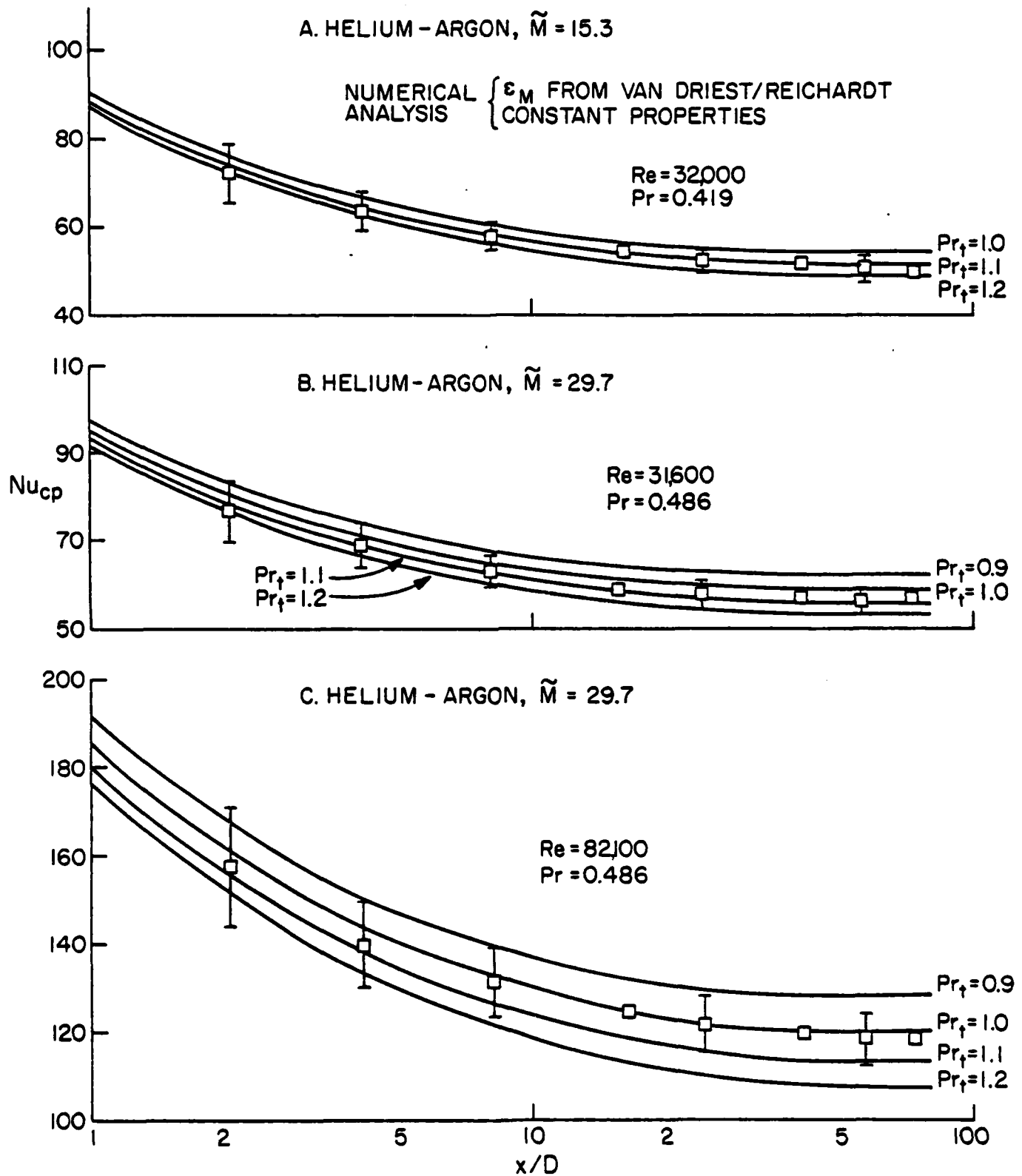


Figure 8. Examples of the Comparisons Between Measured $Nu_{cp}(x)$ and Predicted $Nu(x)$ as Used to Deduce Pr_{tw} for Constant Properties.

but the same Prandtl number. The variation of $Pr_{t,w}$ with respect to Prandtl number was examined by comparing Nu_{cp} at different Prandtl numbers, but the same Reynolds number.

Procedure for Studying High Heating Rates

As mentioned in the Introduction, relatively high heating rates could possibly occur in the heater tubes of the closed Brayton cycle. To calculate Nu_b for helium-argon mixtures at these high heating rates, the numerical method of Bankston and McEligot [42], discussed in the previous section, was used. Properties were allowed to vary, and the relations (9,10) discussed in the Gas Properties section were used. The simple van Driest/Reichardt model eqn. (25), and $Pr_{t,w}$ determined for helium-argon at constant property conditions were incorporated. No radial variation of Pr_t was included, thus, $Pr_t = Pr_{t,w}$. The axial variation of wall heat flux was similar to the one used for the constant property calculations, but was modified slightly for each experimental run to agree with the wall heat flux variation determined from the experimental measurements.

The axial variation of measured Nu_b from two helium-argon experimental runs was compared to the calculated axial variation of Nu_b . From a series of runs with approximately equivalent inlet Reynolds and Prandtl numbers, the runs with the highest and lowest heating rate were

chosen. Since the Pr_t used was for constant properties, one would expect agreement of measured and calculated Nu_b at the low heating rate. If the measured and calculated Nu_b at the high heating rate agreed, this would validate the use of Pr_t determined from constant property conditions for conditions in which properties varied significantly. If the results at the high heating rate did not agree, this might indicate that either, Pr_t determined for constant properties could not be used for variable property conditions, or that some other phenomenon, such as the Soret effect, was acting.

Turbulent Prandtl Number Results and Discussion

Fig. 8 illustrates examples of the comparisons between measured Nu_{cp} and calculated Nu_{cp} used to determine $Pr_{t,w}$. Examples for three Reynolds numbers and two Prandtl numbers are shown. Curves of the calculated Nu_{cp} are included at four different Pr_t (0.9, 1.0, 1.1, 1.2). Brackets indicating the experimental error of the measured Nu_{cp} are also included. Because of the large error in the immediate thermal entry, only results for x/D greater than eight were used to determine $Pr_{t,w}$.

Fig. 8a shows the measured and calculated Nu_{cp} for a helium-argon mixture with a molecular weight of 15.30, Prandtl number of 0.419, and Reynolds number of 32000. By examining results of similar graphs, $Pr_{t,w}$ was determined to be 1.1 ± 0.1 for helium-argon mixtures with molecular

weights of approximately 15, Prandtl numbers of 0.42, and Reynolds numbers between 32000 and 55200. The measured and calculated Nu_{cp} are shown in Fig. 8b and 8c for a helium-argon mixture at a molecular weight of 29.70, Prandtl number of 0.486, and Reynolds numbers of 31600 and 82100. From results of similar graphs, $Pr_{t,w}$ was determined to be 1.0 ± 0.1 for helium-argon mixtures with molecular weights between 27 and 30, Prandtl numbers between 0.46 and 0.49, and Reynolds numbers between 31600 and 102000.

The effect of Reynolds number on $Pr_{t,w}$ can be examined qualitatively using the results in Fig. 8b and 8c. These results are for the same Prandtl number ($Pr = 0.486$), but Reynolds numbers of 31600 and 82100. For x/D greater than eight, and at the low Reynolds number, the measured Nu_{cp} are slightly below the calculated Nu_{cp} for a $Pr_{t,w}$ of 1.0. At the high Reynolds number and same axial length, the measured Nu_{cp} are slightly above the calculated Nu_{cp} for a $Pr_{t,w}$ of 1.0. For the stated conditions, it appears that $Pr_{t,w}$ has a weak dependence on Reynolds number, and decreases slightly as the Reynolds number increases.

The effect of molecular Prandtl number on $Pr_{t,w}$ can be examined using the results from Fig. 8a and 8c summarized in Table 2. The results (24) of McEligot, Pickett and Taylor [16] for air may also be used since the mixture results appear to show that $Pr_{t,w}$ varies only slightly with Reynolds number.

Table 2. Variation of $Pr_{t,w}$ with respect to Prandtl number.

Gas	Molecular Weight	Prandtl Number	$Pr_{t,w}$	Reynolds Number
Helium-argon	15.30	0.419	1.1 ± 0.1	32000
Helium-argon	29.70	0.486	1.0 ± 0.1	31600
Air	28.97	0.72	0.9 ± 0.1	44500

For the range of Prandtl numbers in Table 2, $Pr_{t,w}$ has a relatively strong dependence on Prandtl number and decreases as Prandtl number increases. This dependence agrees with that (equation 4) noted by Reynolds [13].

High Heating Rate Results and Discussion

Fig. 9 shows the results of the measured and calculated axial variation of Nu_b for the two experimental runs that were investigated. The two runs were for a helium-argon mixture at a molecular weight of 29.70, inlet Prandtl number of 0.486, inlet Reynolds numbers of 32000 and 31200, and maximum heating rates of $q^+ = 0.0006$ and $q^+ = 0.0032$. Since at a Pr_t value of 1.0, the measured Nu_{cp} in Fig. 8b were slightly below the calculated Nu_{cp} , a Pr_t value of 1.02 was used. The constants, "a" and "b" in equations (9) and (10) were 0.772 and 0.741, respectively. For both heating rates, the calculated axial variation of Nu_b in Fig. 9 agreed with the measured axial variation of Nu_b , within the accuracy of the measured values. From this example, it appears that $Pr_{t,w}$ determined from constant property results can be used to calculate Nu_b for variable property conditions with heating rates up to, $q^+ = 0.0032$.

At high heating rates a large temperature gradient exists from the wall to the centerline of the tube. At

sufficiently high heating rates, the possibility of separation of the helium and argon due to the Soret effect arises. If separation did occur, Nu_b at a particular axial location would be expected to change since pure helium or argon have higher Prandtl numbers than helium-argon mixtures. For the high heating experimental run in Fig. 9, the largest wall-to-bulk temperature ratios occur in the axial range, $8.1 < x/D < 16.4$. In this axial range, the measured Nu_b do fall slightly above the calculated Nu_b , but this can not necessarily be attributed to the Soret effect, since the calculated Nu_b are within the experimental accuracies of the measured Nu_b .

The effect of high heating on the axial variation of Nu_b can be examined by comparing the low and high heating rate results in Fig. 9. Since the thermal conductivity and viscosity of helium-argon increase as the temperature increases (equations 9,10), this causes Nu_b for high heat flux conditions to be lower than Nu_b for low heat flux conditions. In the immediate thermal entrance region ($x/D < 5$), the small rise in bulk gas temperature has not caused significant bulk property variation, and the Nu_b for the two heating rates are approximately the same. In the fully developed region, the large rise in bulk gas temperature has caused large property variations, and Nu_b are quite different. At $x/D = 57$, Nu_b for $q_{Max}^+ = 0.0032$ is 29 percent lower than Nu_b for $q_{Max}^+ = 0.0006$.

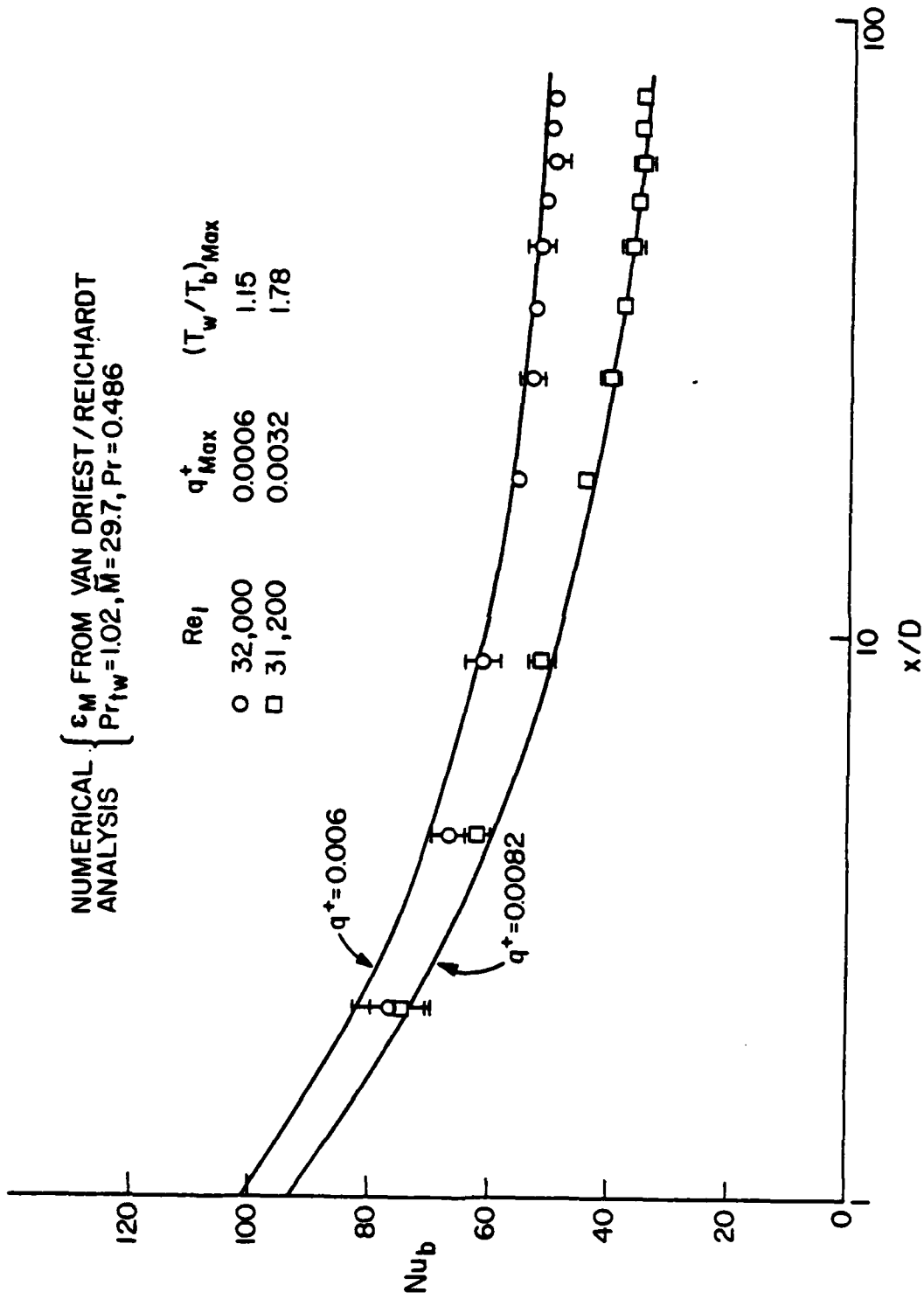


Figure 9. Comparison of Data to Numerical Predictions Accounting for Transport Property Variation.

CONCLUSIONS

The object of this investigation was to study the momentum and heat transfer characteristics for turbulent flow of helium-argon mixtures in tubes. Experimental results were compared to existing experimental correlations, and to results from a numerical analysis. From this investigation the following conclusions have been made:

1. Existing experimental correlations, such as the Drew, Koo, and McAdams relation [34],

$$f = 0.0014 + 0.125 \text{Re}^{-0.32}$$

predict the helium-argon adiabatic friction factors within ± 4.0 percent for turbulent flow in tubes with Reynolds numbers between 31200 and 102000.

2. A correlation suggested by Taylor [36],

$$f = (0.0014 + 0.125 \text{Re}_w^{-0.32}) (T_w/T_b)^{-0.5}$$

predicts average friction factors within ± 10 percent for heated turbulent flow of helium-argon mixtures in tubes with inlet Reynolds numbers between 31200 and 102000.

3. Dittus-Boelter type correlations developed from air and helium experimental data

$$\text{Nu} = 0.021 \text{Re}^{0.8} \text{Pr}^{0.4}$$

overpredict helium-argon Nusselt numbers for constant property, fully developed conditions by as much as 17 percent. An equation of similar form,

$$Nu = 0.021 Re^{0.8} Pr^{0.55}$$

but with the exponent of the Prandtl number changed to 0.55 predicts constant property Nusselt numbers of helium-argon mixtures within ± 5.0 percent. The range of Prandtl numbers was between 0.419 and 0.486, and the range of Reynolds numbers was between 31200 and 102000.

4. For the same range of Reynolds numbers and Prandtl numbers, the entrance and properties variation effects can be accounted for by using the equation

$$Nu_b = 0.021 Re_b^{0.8} Pr_b^{0.55} [(T_w/T_b)^{-0.4} + 0.85 D/x].$$

This equation predicted the bulk Nusselt numbers of helium-argon mixtures within ± 15 percent for x/D between 2.1 and 81.6 and a maximum wall-to-bulk temperature ratio of 1.82.

5. For helium-argon mixtures with molecular weights between 14 and 20, Prandtl numbers of 0.42, Reynolds numbers between 32000 and 55000, and constant property conditions the turbulent Prandtl number in the wall region, $Pr_{t,w}$ was determined to be 1.1 ± 0.1 .

6. For helium-argon mixtures with molecular weights between 27 and 30, Prandtl numbers between 0.46 and 0.49, Reynolds numbers between 32000 and 102000, and constant property conditions $Pr_{t,w}$ was determined to be 1.0 ± 0.1 .

7. For Reynolds numbers between 30000 and 100000, $Pr_{t,w}$ is a weak function of Reynolds number. For the Prandtl number range between 0.42 and 0.72, $Pr_{t,w}$ is

a strong function of Prandtl number, and decreases as Prandtl number increases.

8. At maximum wall heating rates of $q^+ = 0.0032$ ($(T_w/T_b)_{\text{Max}} = 1.78$), $Pr_{t,w}$ determined from constant property conditions can be used in a variable properties numerical analysis to calculate $Nu_b(x)$. For the particular experimental run studied ($Re_i = 31200$, $Pr_i = 0.486$, $Pr_{t,w} = Pr_t = 1.02$), calculated $Nu_b(x)$ agreed with measured $Nu_b(x)$, within the accuracy of the measured values. No separation of the helium-argon mixture was apparent.

APPENDICES

APPENDIX A.
Gas Properties

Helium

Molecular Weight = 4.0026

Specific Heat at Constant Pressure = 1.24036 BTU/LB-R

Temperature (F)	Viscosity (LB/FT-HR)	Conductivity (BTU/HR-FT-F)	Sound Velocity (FT/SEC)
40.00000	4.5339250E-02	8.4353958E-02	3.2161071E+03
70.00000	4.7348801E-02	8.8092741E-02	3.3112430E+03
100.00000	4.9301672E-02	9.1726070E-02	3.4037209E+03
130.00000	5.1202696E-02	9.5262937E-02	3.4937318E+03
160.00000	5.3056299E-02	9.8711577E-02	3.5815203E+03
190.00000	5.4866529E-02	1.0207952E-01	3.6671887E+03
220.00000	5.6637084E-02	1.0537365E-01	3.7509011E+03
250.00000	5.8371342E-02	1.0860025E-01	3.8327355E+03
280.00000	6.0072394E-02	1.1176507E-01	3.9129567E+03
310.00000	6.1743066E-02	1.1487336E-01	3.9915180E+03
340.00000	6.3381257E-02	1.1835159E-01	4.0685326E+03
370.00000	6.5192576E-02	1.2129120E-01	4.1441751E+03
400.00000	6.6752529E-02	1.2419350E-01	4.2184325E+03
430.00000	6.8293346E-02	1.2706020E-01	4.2914552E+03
460.00000	6.9213869E-02	1.2989285E-01	4.3631376E+03
490.00000	7.1320879E-02	1.3269294E-01	4.4337490E+03
520.00000	7.2809100E-02	1.3546179E-01	4.5032339E+03
550.00000	7.4281207E-02	1.3820065E-01	4.5716629E+03
580.00000	7.5737823E-02	1.4091070E-01	4.6399082E+03
610.00000	7.7179543E-02	1.4359302E-01	4.7055364E+03
640.00000	7.8606903E-02	1.4624864E-01	4.7710647E+03
670.00000	8.0020433E-02	1.4887852E-01	4.8357351E+03
700.00000	8.1420607E-02	1.5148355E-01	4.8994128E+03
730.00000	8.2807879E-02	1.5406645E-01	4.9624506E+03
760.00000	8.4182578E-02	1.5662240E-01	5.0246394E+03
790.00000	8.5545409E-02	1.5915777E-01	5.0860581E+03
820.00000	8.6897453E-02	1.6167140E-01	5.1467439E+03
850.00000	8.8236173E-02	1.6416395E-01	5.2067224E+03
880.00000	8.9564910E-02	1.6663600E-01	5.2660179E+03
910.00000	9.0882989E-02	1.6908837E-01	5.3246530E+03
940.00000	9.2190718E-02	1.7152141E-01	5.3826495E+03
970.00000	9.3488391E-02	1.7393574E-01	5.4400277E+03
1000.00000	9.4776287E-02	1.7633188E-01	5.4968070E+03
1030.00000	9.6054671E-02	1.7871032E-01	5.5530057E+03
1060.00000	9.7323797E-02	1.8110715E-01	5.6086414E+03
1090.00000	9.8583908E-02	1.8341538E-01	5.6637306E+03
1120.00000	9.9835234E-02	1.8574400E-01	5.7182390E+03
1150.00000	1.0107800E-01	1.8805625E-01	5.7723319E+03
1180.00000	1.0231241E-01	1.9035230E-01	5.8258734E+03
1210.00000	1.0353867E-01	1.9263433E-01	5.8789273E+03
1240.00000	1.0475693E-01	1.9490102E-01	5.9315068E+03
1270.00000	1.0596752E-01	1.9715323E-01	5.9836242E+03
1300.00000	1.0717047E-01	1.9943013E-01	6.0352215E+03

Helium cont.

Temperature (F)	Enthalpy (BTU/LB)
40.00000	6.1979399E+02
70.00000	6.5700470E+02
100.00000	6.9421541E+02
130.00000	7.3142612E+02
160.00000	7.6863683E+02
190.00000	8.0584754E+02
220.00000	8.4305825E+02
250.00000	8.8026896E+02
280.00000	9.1747967E+02
310.00000	9.5469038E+02
340.00000	9.9190109E+02
370.00000	1.0291118E+03
400.00000	1.0663225E+03
430.00000	1.1035332E+03
460.00000	1.1407439E+03
490.00000	1.1779546E+03
520.00000	1.2151654E+03
550.00000	1.2523761E+03
580.00000	1.2895868E+03
610.00000	1.3267975E+03
640.00000	1.3640082E+03
670.00000	1.4012189E+03
700.00000	1.4384296E+03
730.00000	1.4756403E+03
760.00000	1.5128510E+03
790.00000	1.5500617E+03
820.00000	1.5872725E+03
850.00000	1.6244832E+03
880.00000	1.6616939E+03
910.00000	1.6989046E+03
940.00000	1.7361153E+03
970.00000	1.7733260E+03
1000.00000	1.8105367E+03
1030.00000	1.8477474E+03
1060.00000	1.8849581E+03
1090.00000	1.9221689E+03
1120.00000	1.9593796E+03
1150.00000	1.9965903E+03
1180.00000	2.0338010E+03
1210.00000	2.0710117E+03
1240.00000	2.1082224E+03
1270.00000	2.1454331E+03
1300.00000	2.1826438E+03

Helium-Argon

Molecular Weight = 15.30

Specific Heat at Constant Pressure = 0.32449 BTU/LB-R

Temperature (F)	Viscosity (LB/FT-HR)	Conductivity (BTU/HR-FT-F)	Sound Velocity (FT/SEC)
40.00000	5.2301378E-02	4.0625810E-02	1.6449620E+03
70.00000	5.4834472E-02	4.2473701E-02	1.6936218E+03
100.00000	5.7302310E-02	4.4276174E-02	1.7409220E+03
130.00000	5.9708821E-02	4.6036461E-02	1.7869707E+03
150.00000	6.2057765E-02	4.7757544E-02	1.8318622E+03
190.00000	6.4352726E-02	4.9442169E-02	1.8755795E+03
220.00000	6.5597095E-02	5.1092872E-02	1.9184964E+03
250.00000	6.8794071E-02	5.2711988E-02	1.9603783E+03
280.00000	7.0946667E-02	5.4301672E-02	2.0013840E+03
310.00000	7.3057709E-02	5.5863391E-02	2.0415062E+03
340.00000	7.5119226E-02	5.7479805E-02	2.0809727E+03
370.00000	7.7215247E-02	5.8979570E-02	2.1196467E+03
400.00000	7.9207939E-02	6.0458070E-02	2.1576276E+03
430.00000	8.1169245E-02	6.1916424E-02	2.1949914E+03
450.00000	8.3100944E-02	6.3355619E-02	2.2316510E+03
490.00000	8.5004721E-02	6.4776608E-02	2.2677568E+03
520.00000	8.6888213E-02	6.6180270E-02	2.3032967E+03
550.00000	8.8734662E-02	6.7567427E-02	2.3382965E+03
580.00000	9.0719962E-02	6.8937691E-02	2.3727301E+03
610.00000	9.2501848E-02	7.0372762E-02	2.4067396E+03
640.00000	9.4263621E-02	7.1695393E-02	2.4402858E+03
670.00000	9.5006740E-02	7.3005294E-02	2.4733478E+03
700.00000	9.7731345E-02	7.4302920E-02	2.5059737E+03
730.00000	9.9438497E-02	7.5588637E-02	2.5381302E+03
760.00000	1.0112277E-01	7.6863303E-02	2.5699332E+03
790.00000	1.0280286E-01	7.8126300E-02	2.6013374E+03
820.00000	1.0446140E-01	7.9374854E-02	2.6324367E+03
850.00000	1.0610497E-01	8.0621064E-02	2.6633114E+03
880.00000	1.0773613E-01	8.1853224E-02	2.6939442E+03
910.00000	1.0934943E-01	8.3075648E-02	2.7234330E+03
940.00000	1.1103629E-01	8.4303934E-02	2.7530968E+03
970.00000	1.1269272E-01	8.5506316E-02	2.7824444E+03
1000.00000	1.1422746E-01	8.6699878E-02	2.8114836E+03
1030.00000	1.1578082E-01	8.7884858E-02	2.8402290E+03
1060.00000	1.1732311E-01	8.9051680E-02	2.8686863E+03
1090.00000	1.1885461E-01	9.0229963E-02	2.8968531E+03
1120.00000	1.2037559E-01	9.1390511E-02	2.9247584E+03
1150.00000	1.2188632E-01	9.2543323E-02	2.9524100E+03
1180.00000	1.2333700E-01	9.3688548E-02	2.9797959E+03
1210.00000	1.2487803E-01	9.4826487E-02	3.0069311E+03
1240.00000	1.2635947E-01	9.5957105E-02	3.0338242E+03
1270.00000	1.2783161E-01	9.7080576E-02	3.0604310E+03
1300.00000	1.2929466E-01	9.8197793E-02	3.0869076E+03

Helium-Argon, M = 15.30 cont.

Temperature (F)	Enthalpy (BTU/LB)
40.00000	1.6214297E+02
70.00000	1.7187758E+02
100.00000	1.8161220E+02
130.00000	1.9134681E+02
160.00000	2.0108142E+02
190.00000	2.1081604E+02
220.00000	2.2055065E+02
250.00000	2.3028527E+02
280.00000	2.4001988E+02
310.00000	2.4975449E+02
340.00000	2.5948911E+02
370.00000	2.6922372E+02
400.00000	2.7895833E+02
430.00000	2.8869295E+02
460.00000	2.9842756E+02
490.00000	3.0816217E+02
520.00000	3.1789679E+02
550.00000	3.2763140E+02
580.00000	3.3736602E+02
610.00000	3.4710063E+02
640.00000	3.5683524E+02
670.00000	3.6656986E+02
700.00000	3.7630447E+02
730.00000	3.8603908E+02
760.00000	3.9577370E+02
790.00000	4.0550831E+02
820.00000	4.1524292E+02
850.00000	4.2497754E+02
880.00000	4.3471215E+02
910.00000	4.4444677E+02
940.00000	4.5418138E+02
970.00000	4.6391599E+02
1000.00000	4.7365061E+02
1030.00000	4.8338522E+02
1060.00000	4.9311983E+02
1090.00000	5.0285445E+02
1120.00000	5.1258906E+02
1150.00000	5.2232367E+02
1180.00000	5.3205829E+02
1210.00000	5.4179290E+02
1240.00000	5.5152752E+02
1270.00000	5.6126213E+02
1300.00000	5.7099674E+02

Helium-Argon

Molecular Weight = 15.83

Specific Heat at Constant Pressure = 0.31362 BTU/LB-R

Temperature (F)	Viscosity (LB/FT-HR)	Conductivity (BTU/HR-FT-F)	Sound Velocity (FT/SEC)
40.00000	5.2384618E-02	3.9396159E-02	1.6171303E+03
70.00000	5.4928517E-02	4.1190434E-02	1.6650286E+03
100.00000	5.7407116E-02	4.2940783E-02	1.7115303E+03
130.00000	5.9824291E-02	4.4650306E-02	1.7568015E+03
150.00000	6.2183763E-02	4.6332186E-02	1.8009351E+03
190.00000	6.4489079E-02	4.7958105E-02	1.8440127E+03
220.00000	6.6743604E-02	4.9561496E-02	1.8861067E+03
250.00000	6.8950516E-02	5.1134167E-02	1.9272815E+03
290.00000	7.1112809E-02	5.2678312E-02	1.9675349E+03
310.00000	7.3223329E-02	5.4195807E-02	2.0070387E+03
340.00000	7.5370145E-02	5.5761177E-02	2.0458399E+03
370.00000	7.7405684E-02	5.7219141E-02	2.0838609E+03
400.00000	7.9407606E-02	5.8655777E-02	2.1212006E+03
430.00000	8.1377829E-02	6.0072747E-02	2.1579443E+03
460.00000	8.3318152E-02	6.1471042E-02	2.1939743E+03
490.00000	8.5230263E-02	6.2851537E-02	2.2294706E+03
520.00000	8.7115747E-02	6.4215242E-02	2.2644104E+03
550.00000	8.8976092E-02	6.5562812E-02	2.2988143E+03
590.00000	9.0963336E-02	6.6992607E-02	2.3327207E+03
610.00000	9.2757737E-02	6.8289597E-02	2.3661364E+03
640.00000	9.4527012E-02	6.9537393E-02	2.3990867E+03
670.00000	9.6277033E-02	7.0845900E-02	2.4315306E+03
700.00000	9.8008596E-02	7.2105926E-02	2.4636656E+03
730.00000	9.9722451E-02	7.3335443E-02	2.4953284E+03
760.00000	1.0141931E-01	7.4591815E-02	2.5265345E+03
790.00000	1.0309984E-01	7.5818442E-02	2.5574783E+03
820.00000	1.0476447E-01	7.7034655E-02	2.5879436E+03
850.00000	1.0641441E-01	7.8240817E-02	2.6181532E+03
890.00000	1.0804964E-01	7.9433721E-02	2.6479594E+03
910.00000	1.0967089E-01	8.0624150E-02	2.6774336E+03
940.00000	1.1142182E-01	8.1818356E-02	2.7066166E+03
970.00000	1.1300326E-01	8.2985697E-02	2.7354587E+03
1000.00000	1.1457297E-01	8.4144452E-02	2.7640196E+03
1030.00000	1.1613127E-01	8.5294890E-02	2.7922787E+03
1060.00000	1.1767846E-01	8.6437221E-02	2.8202546E+03
1090.00000	1.1921483E-01	8.7571654E-02	2.8479557E+03
1120.00000	1.2074066E-01	8.8649835E-02	2.8753399E+03
1150.00000	1.2225620E-01	8.9817619E-02	2.9025648E+03
1180.00000	1.2376172E-01	9.0929526E-02	2.9294877E+03
1210.00000	1.2525745E-01	9.2034237E-02	2.9561654E+03
1240.00000	1.2674362E-01	9.3132070E-02	2.9826045E+03
1270.00000	1.2822046E-01	9.4223037E-02	3.0088113E+03
1300.00000	1.2968818E-01	9.5307343E-02	3.0347317E+03

Helium-Argon, M = 15.83 cont.

Temperature (F)	Enthalpy (BTU/LB)
40.00000	1.5671430E+02
70.00000	1.6512300E+02
100.00000	1.7553169E+02
130.00000	1.8494038E+02
160.00000	1.9434907E+02
190.00000	2.0375776E+02
220.00000	2.1316645E+02
250.00000	2.2257514E+02
280.00000	2.3198383E+02
310.00000	2.4139252E+02
340.00000	2.5080121E+02
370.00000	2.6020990E+02
400.00000	2.6961859E+02
430.00000	2.7902728E+02
460.00000	2.8843597E+02
490.00000	2.9784466E+02
520.00000	3.0725335E+02
550.00000	3.1666204E+02
580.00000	3.2607073E+02
610.00000	3.3547942E+02
640.00000	3.4488811E+02
670.00000	3.5429680E+02
700.00000	3.6370549E+02
730.00000	3.7311418E+02
760.00000	3.8252287E+02
790.00000	3.9193156E+02
820.00000	4.0134025E+02
850.00000	4.1074894E+02
880.00000	4.2015763E+02
910.00000	4.2956632E+02
940.00000	4.3897501E+02
970.00000	4.4838370E+02
1000.00000	4.5779239E+02
1030.00000	4.6720108E+02
1050.00000	4.7660977E+02
1090.00000	4.8501846E+02
1120.00000	4.9542715E+02
1150.00000	5.0483584E+02
1180.00000	5.1424453E+02
1210.00000	5.2365322E+02
1240.00000	5.3306191E+02
1270.00000	5.4247060E+02
1300.00000	5.5187929E+02

Helium-Argon

Molecular Weight = 27.53

Specific Heat at Constant Pressure = 0.18034 BTU/LB-R

Temperature (F)	Viscosity (LB/FT-HR)	Conductivity (BTU/HR-FT-F)	Sound Velocity (FT/SEC)
40.00000	5.2563842E-02	2.0434522E-02	1.2263050E+03
70.00000	5.5227557E-02	2.1402132E-02	1.2625304E+03
100.00000	5.7827260E-02	2.2347336E-02	1.2978423E+03
130.00000	6.0365859E-02	2.3271434E-02	1.3321712E+03
160.00000	6.2846258E-02	2.4175647E-02	1.3656374E+03
190.00000	6.5271322E-02	2.5061130E-02	1.3983029E+03
220.00000	6.7643847E-02	2.5928965E-02	1.4302225E+03
250.00000	6.9966537E-02	2.6780169E-02	1.4614451E+03
280.00000	7.2241997E-02	2.7615636E-02	1.4920145E+03
310.00000	7.4472722E-02	2.8436438E-02	1.5219707E+03
340.00000	7.6670221E-02	2.9253647E-02	1.5513471E+03
370.00000	7.8817021E-02	3.0045581E-02	1.5800178E+03
400.00000	8.0926015E-02	3.0825225E-02	1.6084927E+03
430.00000	8.2999207E-02	3.1593220E-02	1.6366317E+03
460.00000	8.5038497E-02	3.2350167E-02	1.6636765E+03
490.00000	8.7045687E-02	3.3096632E-02	1.6905331E+03
520.00000	8.9022431E-02	3.3833145E-02	1.7170378E+03
550.00000	9.0970492E-02	3.4560205E-02	1.7431738E+03
580.00000	9.2984234E-02	3.5333441E-02	1.7688837E+03
610.00000	9.4864927E-02	3.6032088E-02	1.7942259E+03
640.00000	9.6721845E-02	3.6722244E-02	1.8192119E+03
670.00000	9.8556091E-02	3.7405190E-02	1.8438594E+03
700.00000	1.0036871E-01	3.8081200E-02	1.8681317E+03
730.00000	1.0216070E-01	3.8750575E-02	1.8921914E+03
760.00000	1.0393299E-01	3.9413547E-02	1.9159002E+03
790.00000	1.0568648E-01	4.0070366E-02	1.9393192E+03
820.00000	1.0742201E-01	4.0721264E-02	1.9624588E+03
850.00000	1.0914040E-01	4.1366460E-02	1.9853287E+03
880.00000	1.1084239E-01	4.2006162E-02	2.0079381E+03
910.00000	1.1252671E-01	4.2640566E-02	2.0302957E+03
940.00000	1.1450619E-01	4.3315536E-02	2.0524098E+03
970.00000	1.1613465E-01	4.3935632E-02	2.0742382E+03
1000.00000	1.1775107E-01	4.4551214E-02	2.0959382E+03
1030.00000	1.1935579E-01	4.5162402E-02	2.1173569E+03
1060.00000	1.2094912E-01	4.5769313E-02	2.1385308E+03
1090.00000	1.2253134E-01	4.6372057E-02	2.1595364E+03
1120.00000	1.2410274E-01	4.6970738E-02	2.1803396E+03
1150.00000	1.2566359E-01	4.7565458E-02	2.2009361E+03
1180.00000	1.2721415E-01	4.8156313E-02	2.2214116E+03
1210.00000	1.2875466E-01	4.8743339E-02	2.2415411E+03
1240.00000	1.3028537E-01	4.9326736E-02	2.2616897E+03
1270.00000	1.3180649E-01	4.9906596E-02	2.2815021E+03
1300.00000	1.3331826E-01	5.0482880E-02	2.3012929E+03

Helium-Argon, M = 27.53 cont.

Temperature (F)	Enthalpy (BTU/LB)
40.00000	9.0112148E+01
70.00000	9.5522231E+01
100.00000	1.0093231E+02
130.00000	1.0634240E+02
160.00000	1.1175248E+02
190.00000	1.1716256E+02
220.00000	1.2257265E+02
250.00000	1.2798273E+02
280.00000	1.3339281E+02
310.00000	1.3880290E+02
340.00000	1.4421298E+02
370.00000	1.4962306E+02
400.00000	1.5503315E+02
430.00000	1.6044323E+02
460.00000	1.6585331E+02
490.00000	1.7126340E+02
520.00000	1.7667348E+02
550.00000	1.8208356E+02
580.00000	1.8749364E+02
610.00000	1.9290373E+02
640.00000	1.9831381E+02
670.00000	2.0372389E+02
700.00000	2.0913397E+02
730.00000	2.1454406E+02
760.00000	2.1995414E+02
790.00000	2.2536423E+02
820.00000	2.3077431E+02
850.00000	2.3618439E+02
880.00000	2.4159448E+02
910.00000	2.4700456E+02
940.00000	2.5241464E+02
970.00000	2.5782473E+02
1000.00000	2.6323481E+02
1030.00000	2.6864489E+02
1060.00000	2.7405497E+02
1090.00000	2.7946506E+02
1120.00000	2.8487514E+02
1150.00000	2.9028522E+02
1180.00000	2.9569531E+02
1210.00000	3.0110539E+02
1240.00000	3.0651547E+02
1270.00000	3.1192556E+02
1300.00000	3.1733564E+02

Helium-Argon

Molecular Weight = 29.70

Specific Heat at Constant Pressure = 0.16716 BTU/LB-R

Temperature (F)	Viscosity (LB/FT-HR)	Conductivity (BTU/HR-FT-F)	Sound Velocity (FT/SEC)
40.00000	5.2418661E-02	1.5071935E-02	1.1806560E+03
70.00000	5.5089766E-02	1.8936920E-02	1.2155311E+03
100.00000	5.7697317E-02	1.9781982E-02	1.2495304E+03
130.00000	6.0244079E-02	2.0608217E-02	1.2825314E+03
160.00000	6.2732833E-02	2.1416666E-02	1.3148018E+03
190.00000	6.5166339E-02	2.2208326E-02	1.3462513E+03
220.00000	6.7547299E-02	2.2984125E-02	1.3769827E+03
250.00000	6.9878341E-02	2.3744946E-02	1.4070430E+03
280.00000	7.2162005E-02	2.4491622E-02	1.4364745E+03
310.00000	7.4400728E-02	2.5224934E-02	1.4653149E+03
340.00000	7.6602596E-02	2.5952050E-02	1.4935385E+03
370.00000	7.8757404E-02	2.6659758E-02	1.5213564E+03
400.00000	8.0874035E-02	2.7356255E-02	1.5486168E+03
430.00000	8.2954484E-02	2.8042132E-02	1.5754055E+03
460.00000	8.5000641E-02	2.8717933E-02	1.6017465E+03
490.00000	8.7014303E-02	2.9384180E-02	1.6275511E+03
520.00000	8.8997171E-02	3.0041350E-02	1.6531695E+03
550.00000	9.0950857E-02	3.0669903E-02	1.6782302E+03
580.00000	9.2853308E-02	3.1376422E-02	1.7030405E+03
610.00000	9.4841908E-02	3.1999398E-02	1.7274352E+03
640.00000	9.6705728E-02	3.2615263E-02	1.7514321E+03
670.00000	9.8546407E-02	3.3224512E-02	1.7752220E+03
700.00000	1.0036503E-01	3.3827417E-02	1.7986390E+03
730.00000	1.0216262E-01	3.4424234E-02	1.8217549E+03
760.00000	1.0394016E-01	3.5015209E-02	1.8445812E+03
790.00000	1.0569856E-01	3.5600570E-02	1.8671254E+03
820.00000	1.0743387E-01	3.6180536E-02	1.8894066E+03
850.00000	1.0916143E-01	3.6755321E-02	1.9114352E+03
880.00000	1.1086752E-01	3.7325114E-02	1.9331329E+03
910.00000	1.1255773E-01	3.7890104E-02	1.9547183E+03
940.00000	1.1456435E-01	3.8502097E-02	1.9760392E+03
970.00000	1.1619396E-01	3.9053187E-02	1.9970732E+03
1000.00000	1.1781154E-01	3.9600222E-02	2.0179173E+03
1030.00000	1.1941741E-01	4.0143343E-02	2.0385482E+03
1060.00000	1.2101187E-01	4.0682804E-02	2.0589725E+03
1090.00000	1.2259524E-01	4.1218466E-02	2.0791361E+03
1120.00000	1.2416778E-01	4.1750517E-02	2.0992249E+03
1150.00000	1.2572976E-01	4.2279046E-02	2.1190544E+03
1180.00000	1.2728145E-01	4.2804138E-02	2.1387199E+03
1210.00000	1.2882309E-01	4.3325877E-02	2.1581364E+03
1240.00000	1.3035492E-01	4.3844340E-02	2.1774987E+03
1270.00000	1.3187716E-01	4.4359603E-02	2.1966314E+03
1300.00000	1.3339004E-01	4.4871734E-02	2.2155988E+03

Helium-Argon, M = 29.70 cont.

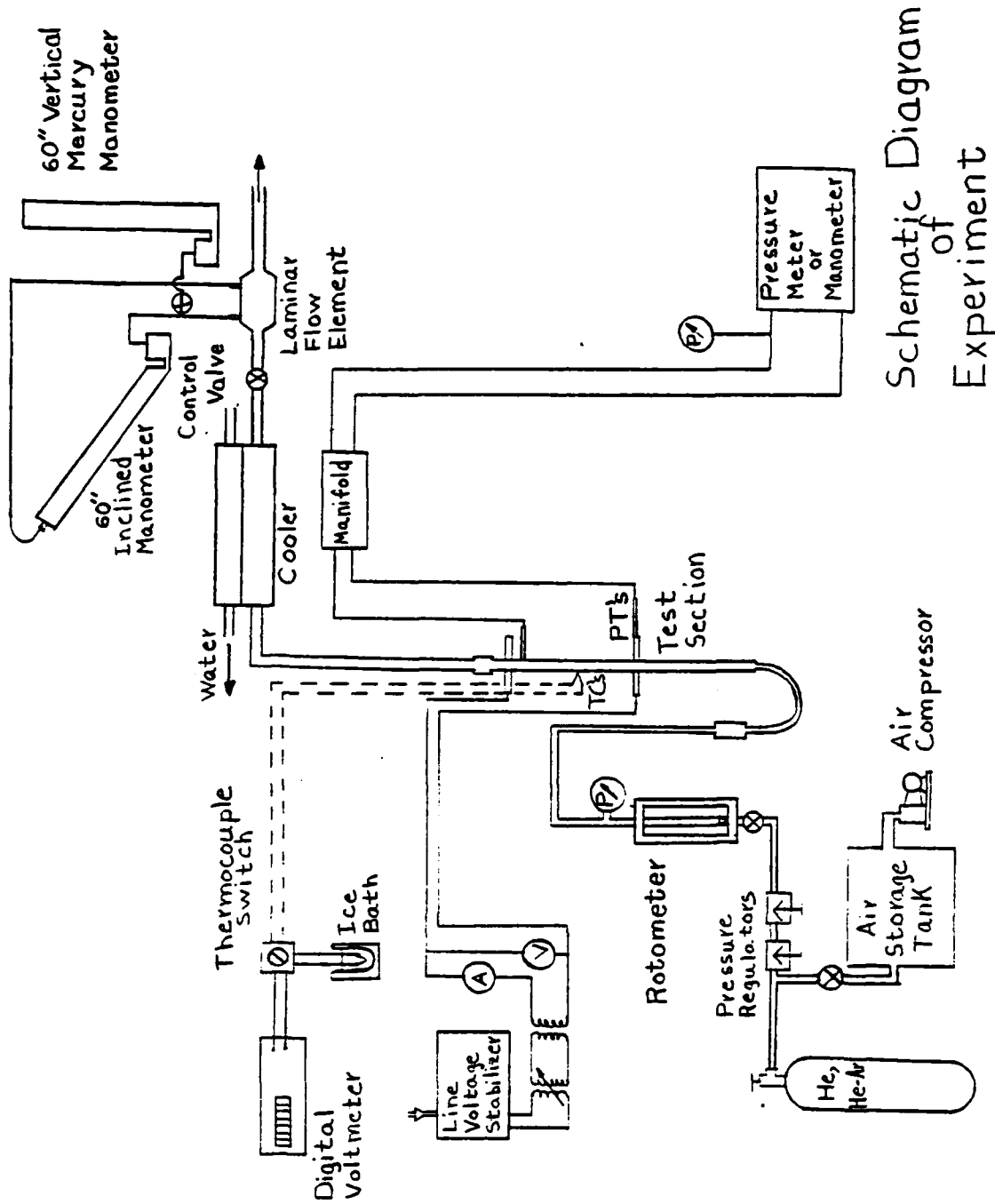
Temperature (F)	Enthalpy (BTU/LB)
40.00000	8.3528196E+01
70.00000	8.8542997E+01
100.00000	9.3557798E+01
130.00000	9.8572599E+01
160.00000	1.0358740E+02
190.00000	1.0850220E+02
220.00000	1.1361700E+02
250.00000	1.1863140E+02
280.00000	1.2364660E+02
310.00000	1.2856141E+02
340.00000	1.3357621E+02
370.00000	1.3859101E+02
400.00000	1.4370581E+02
430.00000	1.4872051E+02
460.00000	1.5373541E+02
490.00000	1.5875021E+02
520.00000	1.6376501E+02
550.00000	1.6877981E+02
580.00000	1.7379461E+02
610.00000	1.7880941E+02
640.00000	1.8382422E+02
670.00000	1.8883902E+02
700.00000	1.9385382E+02
730.00000	1.9886862E+02
760.00000	2.0388342E+02
790.00000	2.0889822E+02
820.00000	2.1391302E+02
850.00000	2.1892782E+02
880.00000	2.2394262E+02
910.00000	2.2895742E+02
940.00000	2.3397222E+02
970.00000	2.3898702E+02
1000.00000	2.4400183E+02
1030.00000	2.4901663E+02
1060.00000	2.5403143E+02
1090.00000	2.5904623E+02
1120.00000	2.6406103E+02
1150.00000	2.6907583E+02
1180.00000	2.7409063E+02
1210.00000	2.7910543E+02
1240.00000	2.8412024E+02
1270.00000	2.8913504E+02
1300.00000	2.9414984E+02

APPENDIX B. EXPERIMENT

Apparatus

A schematic diagram of the experiment is shown in Fig. B1. The helium and helium-argon mixtures were bought from manufacturers in high pressure bottles. The air was obtained from a large storage tank that was replenished by a compressor. Two regulators were used to reduce and stabilize the pressure. A Brooks rotometer was used to obtain a rough measurement of the flow rate. A Bourdon tube Heise gage measured the pressure just downstream of the rotometer. A small tank constructed to mix the gas, and instrumented with a thermocouple measured the inlet stagnation temperature. A sketch of the test section from the inlet to just below the outlet mixing tank, and displaying the location of thermocouples, pressure taps, electrodes, and voltage taps is shown in Fig. B2.

Power was measured using a Fluke differential voltmeter and Weston ammeter in the same manner as Perkins et al. [27]. Whenever possible the power supply described by Perkins et al. [27] was used. When it did not supply sufficient power an a.c. Lincoln welder (Model TM-500/500) was used. To determine a power factor when the welder was used, the power measured with a Weston watt meter was compared to that calculated from voltage (Fluke voltmeter)



Schematic Diagram
of
Experiment

Fig. B1.

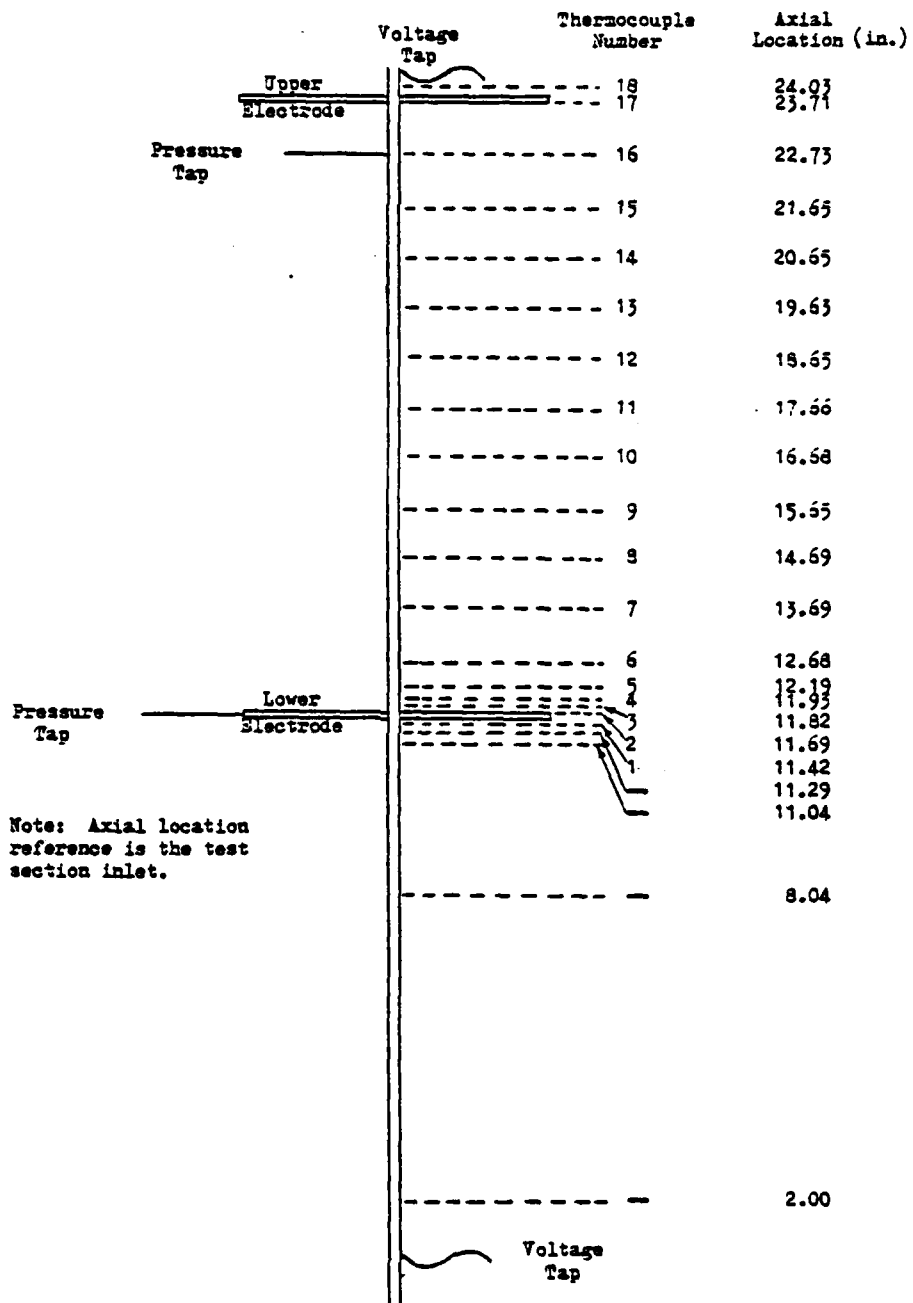


Fig. 32. Diagram of test section.

and current (Weston ammeter) measurements. A power factor of 1.0 was used since the two measurements agreed within 3.3 percent.

Thermocouple output was measured with a Hewlett Packard digital voltmeter. An ice bath was used as a reference for all of the thermocouples. Thermocouples were selected for measurement using a manual switch. The numbered thermocouples on Fig. B2 were used in the computer program that reduced the experimental data for the heated runs. This computer program is described by a number of people [30,31,32,44]. The unnumbered thermocouples were used to determine the amount of preheating of the gas before it entered the heated section.

Pressure drop in the test section was measured with Meriam 60 inch vertical water or mercury manometers. Inlet static pressure was measured with a Bourdon tube Heise gage.

After the gas passed through the test section, it was cooled by a chilled water counterflow heat exchanger. The valve used to control the flow rate was located just downstream of the heat exchanger. The heat exchanger was necessary in order that a Meriam laminar flow element could be used. The laminar flow element was used to obtain an accurate measurement of the flow rate. A Meriam 60 inch inclined water manometer with a 10 inch range was used to measure pressure drop across the laminar flow

element. The temperature of the gas in the flow element was measured with a thermocouple, and the pressure was measured with a Meriam 60 inch vertical mercury manometer. The accuracies of the instruments used in this investigation are listed in appendix C (Table C1).

Procedure

Before any experimental runs with gas flow were performed, the test section was heated without flow in order that the heat lost to the environment and the resistance of the test section could be determined. These items are discussed in detail in the following two sections.

The system was purged and all of the instruments zeroed before each set of experimental runs. The purging was done by pressurizing the system to approximately 100 psig with the gas to be used. The system was then allowed to blow down to approximately 10 psig. This sequence was performed four times.

The desired inlet Reynolds number was established by adjusting the pressure level and mass flow rate. Before power was supplied to the test section, measurements were taken so that calculation of the adiabatic friction coefficient was possible. The measured adiabatic friction coefficients were compared to the Drew, Koo, and McAdams correlation [34] (equation 11), and agreement ensured that the pressure measurements, mass flow rate measurements, and mixture molecular weights were correct.

The test section was then heated to the desired level. Since a small period of time elapsed while the thermocouples were manually recorded, pressure drop, pressure level, voltage, current, and mass flow rate measurements were taken before and after the thermocouple measurements. The average of the two measurements was used for data reduction. The inlet Reynolds number was maintained approximately constant while the test section wall temperature was varied by varying the power input. Measurements were taken for a number of different power inputs.

Heat Loss Calibration

In order to calculate the heat transfer coefficient, the heat addition to the gas, q'_{gas} , must be determined. If an energy balance for a small section of the tube is performed, the result is:

$$q'_{\text{gas}} = q'_{\text{gen}} - (q'_{\text{cond}} + q'_{\text{loss}}). \quad (\text{B1})$$

The heat generated in the small section of the tube, q'_{gen} , is:

$$q'_{\text{gen}} = I^2 R'. \quad (\text{B2})$$

The current was measured, and the calculation of the resistance per unit length is discussed in the next section. The axial heat loss due to conduction is:

$$q'_{\text{cond}} = -K A_{\text{cs}} d^2 T_w / dx^2. \quad (\text{B3})$$

The second derivative was determined using a numerical parabolic fit described by McEligot [31]. The variation of thermal conductivity with temperature for Hastelloy-X

was determined from data supplied by the manufacturer [46].

$$K = [5.1 + (0.00622)(T)](\text{Btu/hr-ft-F}) \quad (\text{B4})$$

(T in degrees Fahrenheit)

To determine the heat loss, q'_{loss} , the test section was heated at different levels without gas flow. A program described by Coon [32] was used to calculate the heat loss at each thermocouple. The heat loss was determined as a function of the tube wall and environment temperature difference. The environment temperature was measured with a thermocouple a few inches away from the test section. Figures B3, B4, B5, and B6 show the results for each thermocouple (thermocouple 4 is not included). Except for thermocouple three and four, the data for each thermocouple was fitted with an equation of the form:

$$q'_{\text{loss}} = C_1(T_w - T_\infty) + C_2(T_w - T_\infty)^2 + C_3(T_w - T_\infty)^3 \quad (\text{Btu/hr-ft}) \quad (\text{B5})$$

($T_w - T_\infty$ in degrees Fahrenheit)

The numerical values of C_1 , C_2 , and C_3 are listed below.

Thermocouple	C_1	C_2	C_3
2	2.43E+00	9.43E-03	-2.53E-05
5	1.22E-01	1.84E-04	-1.64E-08
6	9.83E-02	2.30E-04	-6.57E-08
7	8.84E-02	1.66E-04	-3.79E-09
8	8.30E-02	1.35E-04	3.07E-08
9	8.16E-02	1.03E-04	6.04E-08
10	7.89E-02	1.12E-04	4.96E-08
11	7.70E-02	8.82E-05	7.13E-08
12	7.40E-02	1.03E-04	5.50E-08
13	7.54E-02	8.08E-05	7.59E-08
14	7.31E-02	9.77E-05	5.73E-08
15	8.09E-02	8.07E-05	7.46E-08
16	1.18E-01	1.59E-04	5.59E-08
17	2.76E-01	1.14E-03	1.19E-06

The data for thermocouple three was fitted with a straight line determined using the method of least squares.

K&E 10 X 10 TO THE CENTIMETER 46 1513
10 X .25 CM.
KEUFFEL & ESSER CO.

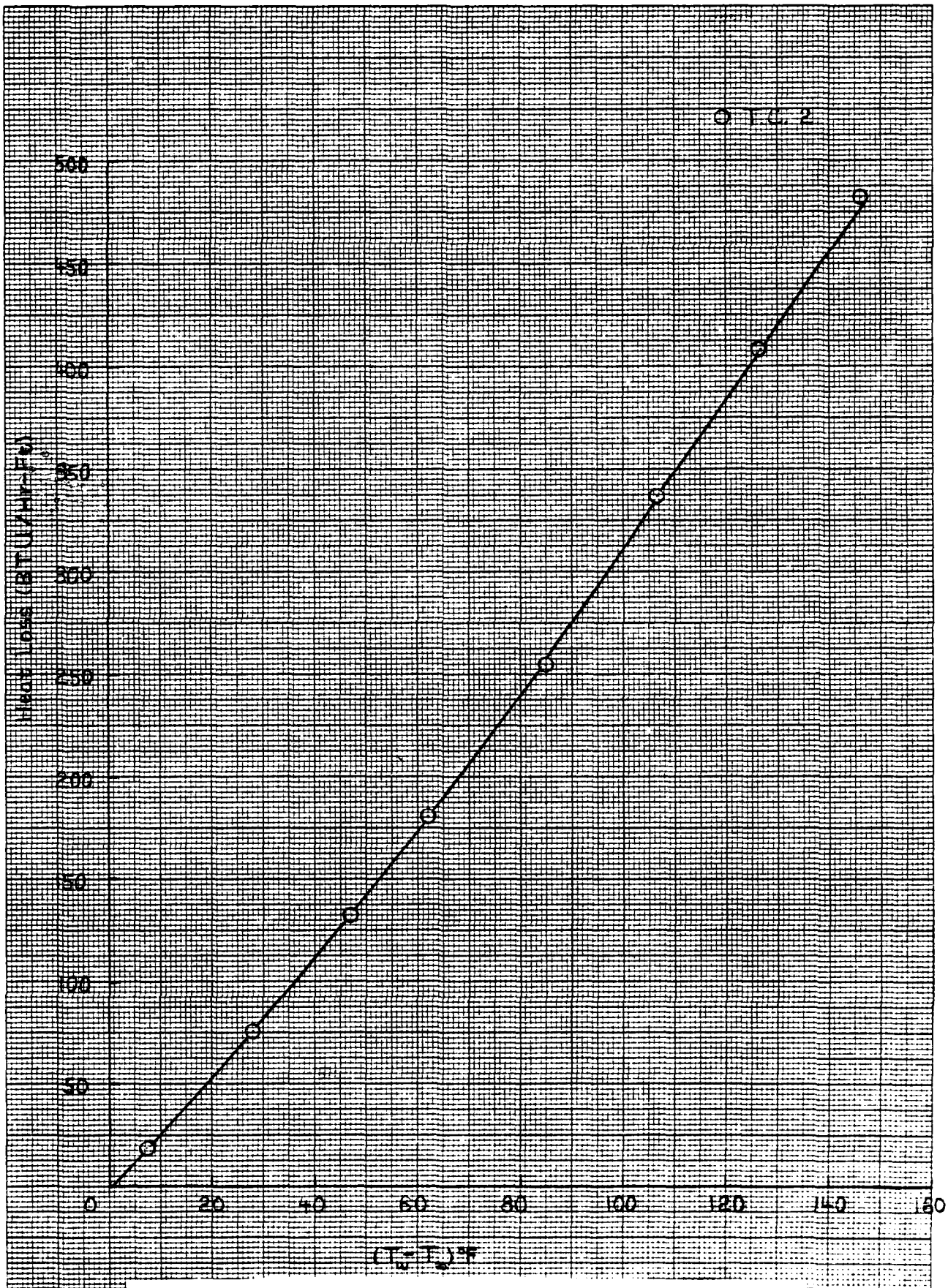


Fig. B3. Heat loss calibration for thermocouple 2.

K&E 10 X 10 TO THE CENTIMETER 46 1513
 1.8 X .25 CM. MADE IN U.S.A.
 KEUFFEL & ESSER CO.

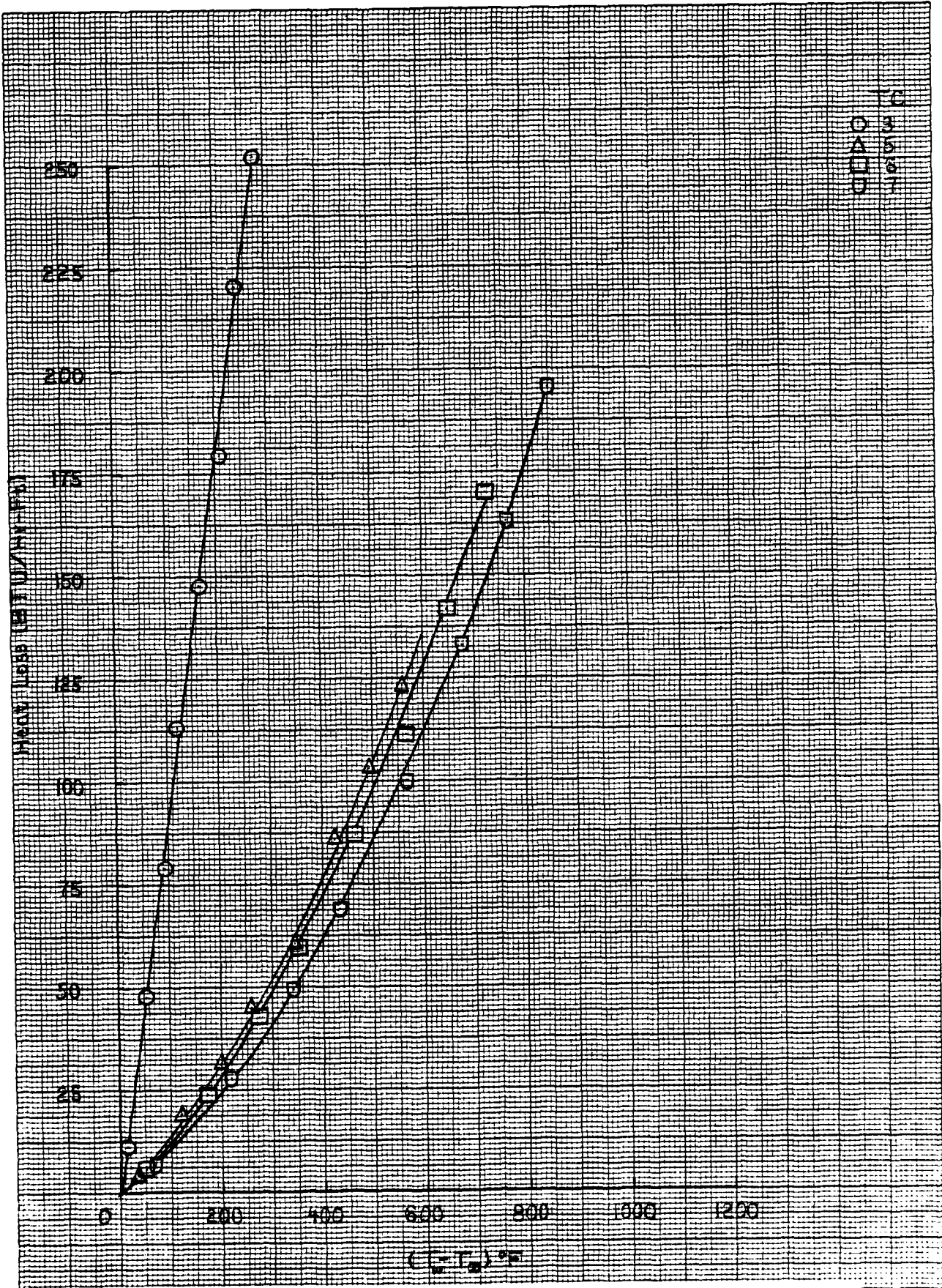


Fig. B4. Heat loss calibration for thermocouples 3, 5, 6, and 7.

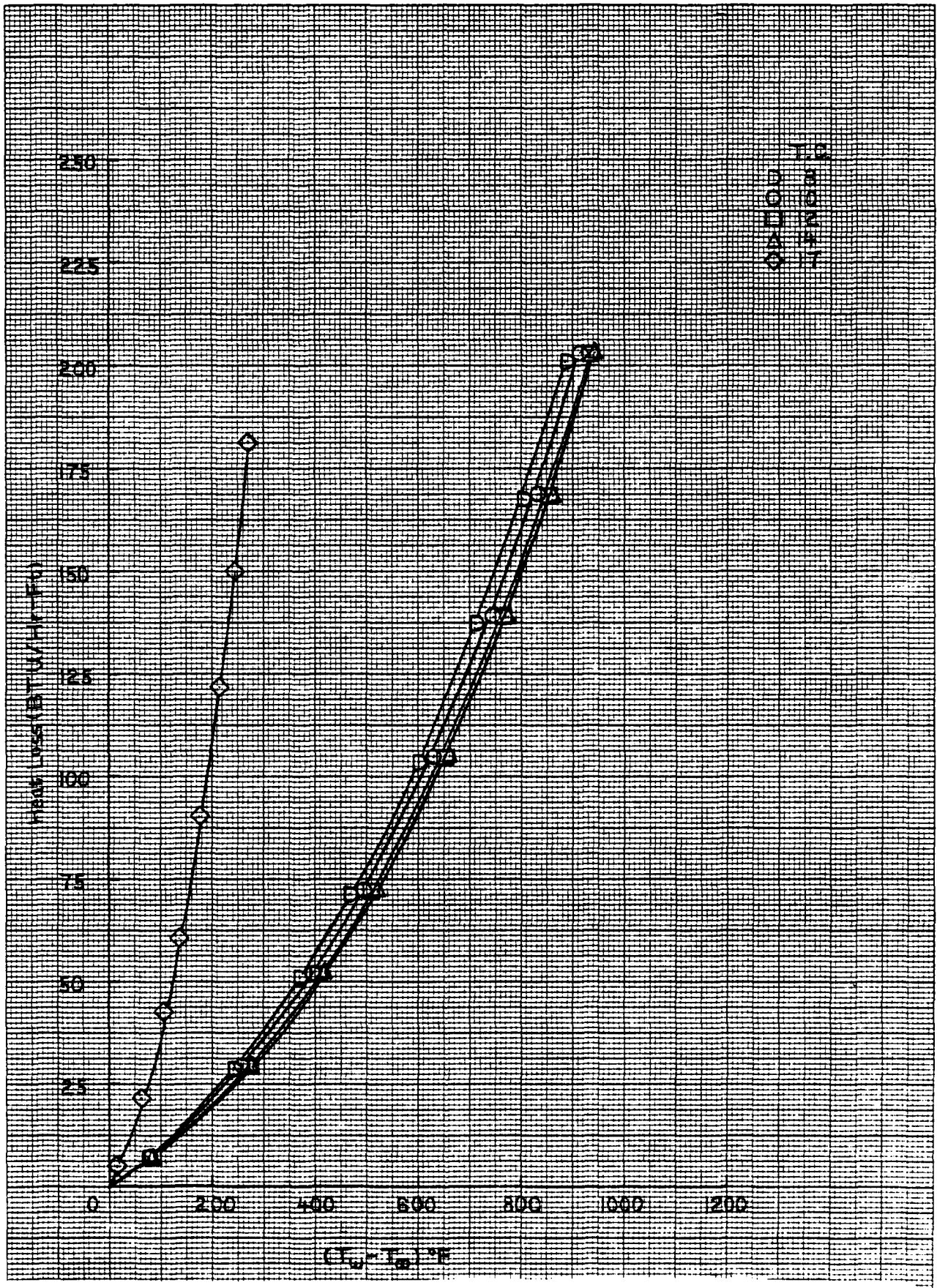


Fig. B5. Heat loss calibration for thermocouples 8, 10, 12, 14, and 17.

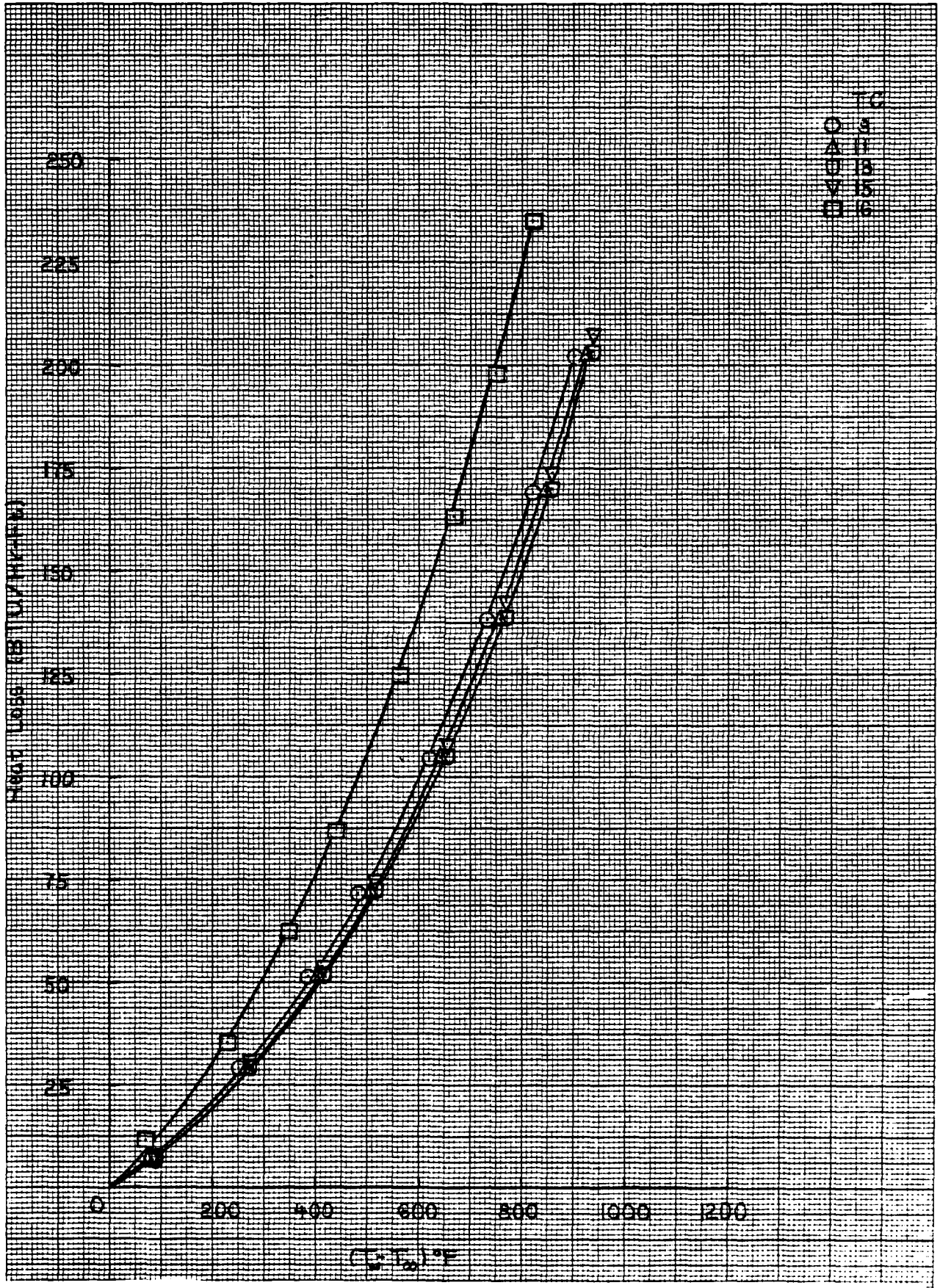


Fig. B6. Heat loss calibration for thermocouples 9, 11, 13, 15, and 16.

$$q'_{\text{loss}}(\text{T.C. 3}) = 0.91(T_w - T_\infty) - 2.20(\text{Btu/hr-ft}) \quad (\text{B6})$$

($T_w - T_\infty$ in degrees Fahrenheit)

The data for thermocouple four was scattered and a representative curve could not be fitted. For this reason the heat loss at this thermocouple was neglected. This introduced only a small error since the heat loss for thermocouple four, even at the highest heating rates, was less than five percent of the heat added to the gas. The fitted curves are also shown on Figures B3, B4, B5, and B6.

Measurement of Test Section Resistance

The variation of resistance with temperature was measured by heating the test section without gas flow in the same manner as was done for the heat loss calibration. Using the thermocouple wires as voltage taps, a measurement of the voltage drop between thermocouple 14 and the lowermost thermocouple on the tube was taken. Another measurement of the voltage drop between thermocouple 12 and the lowermost thermocouple was taken. The difference between these two measurements gave the voltage drop between thermocouples 12 and 14. The section of the tube between thermocouples 12 and 14 was used since the wall temperature for this length was approximately constant.

For a particular power setting the voltage drop discussed in the previous paragraph, the current, and the average of the temperatures at thermocouples 12, 13, and

14 were recorded. Using these measurements the resistance per unit length was determined as a function of temperature. The results are shown on Fig. B7. Also shown on Fig. B7 is the line that was used to approximate the variation of resistance with temperature.

$$R' = [3.98 \times 10^{-4}(T) + 4.745] \text{ (m}\Omega\text{/in)} \quad (B7)$$

(T in degrees Fahrenheit)

Meriam Laminar Flow Element Calibration

The laminar flow element was calibrated using a Parkinson-Cowan Type D1 positive displacement flow meter as a standard. It is specified to have 1/2 percent accuracy at ambient conditions and was calibrated by Tucson Gas and Electric before being used. Meriam [45] suggests the following equation for the laminar flow element:

$$\Delta P = A'Q\mu L/D^4 + B'\rho Q^2/D^4. \quad (B8)$$

A' and B' are the constants to be determined by calibration. Since the length, L, and the hydraulic diameter, D, of the laminar flow element passages remain constant, they can be incorporated into new calibration constants, A and B.

$$A = A'L/D^4 \quad B = B'/D^4$$

Equation B8 now becomes:

$$\Delta P = A Q \mu + B \rho Q^2. \quad (B9)$$

If this equation is solved for Q the result is:

$$Q = [-A\mu/B + \sqrt{(A\mu/B)^2 + 4\rho\Delta P/B}]/2\rho. \quad (B10)$$

If both sides of this equation are multiplied by the density, ρ , the result is:

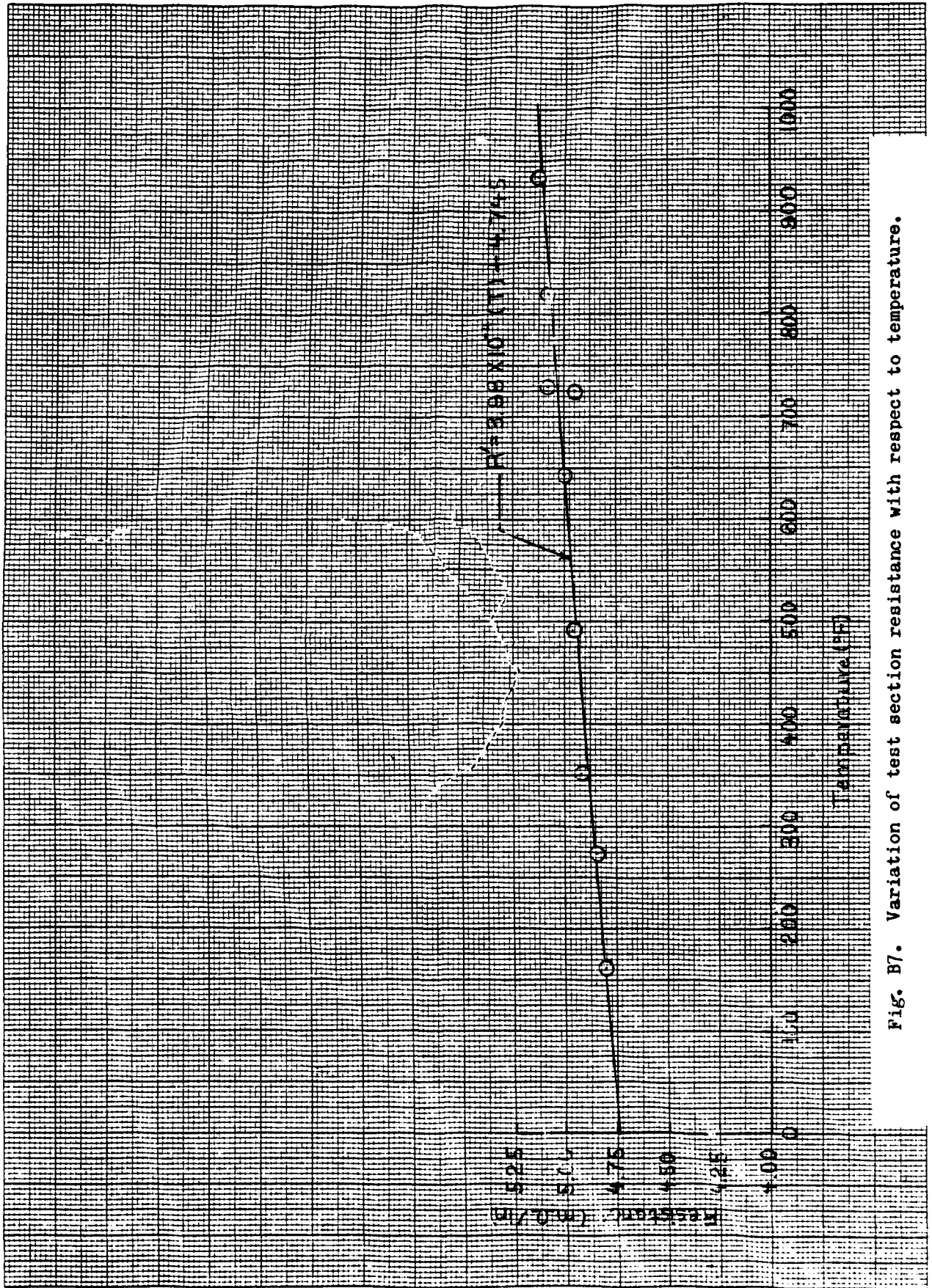


Fig. B7. Variation of test section resistance with respect to temperature.

$$\dot{m} = \rho Q = 1/2[-Au/B + \sqrt{(Au/B)^2 + 4\rho\Delta P/B}]. \quad (B11)$$

Mass flow rate measurements were taken simultaneously with the positive displacement flow meter and the laminar flow element. Both air and helium were used. These gases bound the range of helium-argon mixture molecular weights that were used. From these measurements, A and B were determined so that the maximum difference between the positive displacement flow meter measurements and the laminar flow element measurements was 1.5 percent. The numerical values of A and B were 1375 and 672, respectively, if the parameters in equation B11 have the following units.

m - lb/sec
μ - lb/hr-ft
ρ - lb/ft³
ΔP - inches of water

APPENDIX C
Uncertainty Analysis

An analysis to determine the uncertainty of the results calculated from the measured experimental data was performed. The uncertainties of the directly measured quantities were determined from manufacturers' specifications and experience. Table C1 lists the uncertainties of the instruments used in this investigation. The error propagated to the calculated results from the uncertainties in the measured quantities was determined using a method described by Bottaccini [47]. The general equation used was

$$\begin{aligned} \sigma_Z^2 = & \left(\frac{\partial Z}{\partial Y_1}\right)^2 \sigma_{Y_1}^2 + \left(\frac{\partial Z}{\partial Y_2}\right)^2 \sigma_{Y_2}^2 + \dots \\ & + \left(\frac{\partial Z}{\partial a_1}\right)^2 \sigma_{a_1}^2 + \left(\frac{\partial Z}{\partial a_2}\right)^2 \sigma_{a_2}^2 + \dots \end{aligned} \quad (C1)$$

σ_x is the variance or standard deviation of the xth quantity. Z is the calculated quantity, and Y_i and a_i are the measured values and system parameters used to calculate Z.

To illustrate the above technique, a simple example will be done. The power supplied to the tube can be determined using the relation (assuming a power factor of one)

$$P = E I. \quad (C2)$$

If equation C1 is applied, the error or variance in the power caused by uncertainties in the voltage and current measurements is

$$\sigma_P = \sqrt{(I)^2 \sigma_E^2 + (E)^2 \sigma_I^2} \quad (C3)$$

The result can be presented in the following form so that the percent of uncertainty can easily be determined.

$$\sigma_P/P = \sqrt{(\sigma_E/E)^2 + (\sigma_I/I)^2} \quad (C4)$$

TABLE C1
Uncertainties of Measured Values

Measured quantity	Instrument	Uncertainty
Current	Weston 370 AC/DC ammeter	±0.25% of full scale
Voltage	Fluke 883AB differential voltmeter	±0.1% of input
Mass flow rate	Meriam 50MH10-1 laminar flow element	±1.5% of flow rate
Wall and inlet bulk temperature	Premium grade chromel-alumel thermocouples	±2°F, 3/8% of reading above 535°F
Thermocouple location Pressure tap location	Gaertner M911 Cathetometer	±0.1mm
Diameter	Manufacturers' specifications	±0.001 in.
Pressure	12 inch Heise gage	±0.15 psi
Pressure drop	60" Meriam 30EB25 vertical H ₂ O manometer 60" Meriam 30EB25 vertical Hg manometer	±0.05 in-H ₂ O ±0.05 in-Hg

Table C2 lists the percentage uncertainty in the measured bulk Nusselt numbers for two representative helium-argon runs. The dominant uncertainty in the bulk Nusselt number is the bulk stagnation temperature. For convenience no uncertainty was included for the gas properties. The values used were assumed to be precise.

TABLE C2
 Percentage Uncertainties in the Measured
 Bulk Nusselt Numbers of Helium-Argon

Run	126H	131H
Molecular weight	15.30	15.30
Re _i	56100	54700
(T _w ⁱ /T _b) MAX	1.17	1.77

x/D	Percentage Uncertainty	
1.17	13	10
2.07	8	6
4.14	6	4
8.13	5	3
24.52	4	3
40.79	4	3
56.88	4	3

APPENDIX D
Helium-Argon Experimental Data

The headings and their definitions used in the listing
of the adiabatic friction data are below.

<u>Heading</u>	<u>Definition</u>
Run	Experiment run number
Date	Date on which experimental run was made
Gas	Gas used in the experiment
Molec. wt.	Molecular weight
T_1	Inlet mixer temperature
\dot{m}	Gas flow rate
Re_1	Inlet Reynolds number
P_1	Static pressure at inlet pressure tap
P_2	Static pressure at outlet pressure tap
Static Mach ₁	Static mach number at inlet pressure tap
Static Mach ₂	Static mach number at outlet pressure tap
f_{ad}	Adiabatic friction factor

Table D-1 HELIUM-ARGON ADIABATIC FRICTION FACTOR DATA

Run	Date	Gas	Molec. Wt.	T _i (°F)	ṁ (lb/hr)	Re _i	P ₁ (psia)	P ₂ (psia)	Static Mach ₁	Static Mach ₂	f _{ad}
77A	8/27/75	He	4.003	71.5	3.8	9950	105.4	104.7	0.0521	0.0524	0.00796
81A	8/30/75	He	4.003	72.7	3.7	9660	102.0	101.3	0.0524	0.0527	0.00817
83A	8/30/75	He	4.003	78.0	12.7	32900	80.1	71.6	0.228	0.255	0.00582
85A	9/10/75	He	4.003	75.5	12.5	32500	76.4	67.6	0.235	0.265	0.00586
86A	9/12/75	He-Ar	15.83	72.3	12.3	27800	97.7	96.1	0.0916	0.0931	0.00619
87A	9/12/75	He-Ar	15.83	72.7	24.6	55400	76.0	68.6	0.233	0.258	0.00509
88A	10/ 8/75	He-Ar	15.83	73.0	26.5	59700	74.6	65.4	0.255	0.290	0.00515
91A	10/ 8/75	He-Ar	15.83	72.7	27.7	62400	83.0	74.4	0.240	0.267	0.00503
92A	10/13/75	He-Ar	15.83	72.3	14.1	31700	95.8	93.8	0.106	0.109	0.00607
95A	10/13/75	He-Ar	15.83	72.7	14.1	31800	95.5	93.5	0.107	0.110	0.00597
96A	10/20/75	He-Ar	15.83	72.7	38.3	86100	96.3	82.3	0.285	0.331	0.00468
99A	11/17/75	He-Ar	29.70	70.0	41.8	94200	74.5	63.1	0.293	0.344	0.00458
100A	11/17/75	He-Ar	29.70	70.0	29.9	67300	91.7	87.3	0.172	0.180	0.00493
101A	1/28/76	He-Ar	29.70	71.0	37.0	83200	110.5	105.5	0.176	0.185	0.00447
106A	1/28/76	He-Ar	29.70	72.7	13.9	31100	93.0	91.9	0.0790	0.0799	0.00614
110A	1/29/76	He-Ar	29.70	67.0	14.3	32400	93.4	92.2	0.0810	0.0819	0.00585
113A	2/ 6/76	He-Ar	27.53	73.0	13.5	30200	89.8	88.7	0.0828	0.0838	0.00588
114A	2/ 6/76	He-Ar	27.53	74.0	34.7	77500	110.2	105.0	0.173	0.181	0.00476
115A	2/21/76	He-Ar	27.53	70.5	43.8	98300	96.7	87.1	0.247	0.273	0.00457
121A	2/21/76	He-Ar	27.53	71.0	43.6	97800	95.3	85.5	0.249	0.277	0.00459
122A	3/ 1/76	He-Ar	15.30	70.5	24.7	55700	77.5	70.2	0.232	0.256	0.00497
123A	3/ 1/76	He-Ar	15.30	70.5	14.2	32200	92.8	90.7	0.113	0.115	0.00568

The headings and their definitions used in the listing of the heated flow data are below. The headings that are self-explanatory, or that were used in the listing of the adiabatic friction data are not included.

<u>Heading</u>	<u>Definition</u>
TIN	Inlet mixer temperature
TOUT	Outlet mixer temperature
I	Alternating current
E	Voltage drop between voltage taps
TC	Thermocouple number
X/D	Corresponds to x/D in text
TW	Inside tube wall temperature
TW/TB	Wall-to-bulk temperature ratio
QGAS	Wall heat flux
Q^+	Non-dimensional heat flux parameter. Corresponds to q^+ in text.
PT	Pressure tap: 1-inlet, 2-outlet
TB	Bulk static temperature

RUN 99HP, DATE 10/08/75, GAS HE-AR, MOLECULAR WT. = 15.83
 PIN = 72.9 P, TOUT = 308.7 P, MASS FLOW RATE = 24.1 LB/HR, I = 100.2 AMPS, E = 5.765 VOLTS
 PR,IM = .415, GR/RESQ = .725E-04, MACH(2) = .217, MACH(16) = .305, T,SURR = 75.5 P

IC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSLEIT	QGAS BTU/HR/FT ²	2*
2	.1	130.0	1.122	34772.	.062	250.36	57729.3	.JJ1195
3	1.2	194.6	1.239	54528.	.105	116.28	55763.4	.JJ1231
4	2.1	224.4	1.290	54334.	.030	101.37	59976.7	.JJ1245
5	4.1	256.0	1.336	53891.	.020	46.42	50655.1	.JJ1251
6	9.1	297.5	1.369	53073.	.018	76.37	50965.9	.JJ1256
7	15.4	319.1	1.374	51479.	.016	71.00	61182.5	.JJ1257
8	24.5	348.4	1.375	50042.	.018	56.62	61233.2	.JJ1253
9	32.4	366.7	1.360	48742.	.017	65.88	61329.4	.JJ1253
10	40.8	390.0	1.351	47451.	.019	63.36	61308.0	.JJ1253
11	49.7	407.1	1.336	46315.	.019	53.70	61400.4	.JJ1253
12	59.9	430.6	1.330	45229.	.021	51.64	61359.4	.JJ1253
13	70.3	448.1	1.318	44229.	.022	59.42	61417.9	.JJ1253
14	81.1	472.0	1.313	43249.	.025	59.52	61349.9	.JJ1253
15	91.3	487.3	1.298	42351.	.027	60.07	61282.7	.JJ1253
16	99.1	493.3	1.269	41452.	.084	63.05	61274.0	.JJ1237
17	99.0	435.6	1.165	40769.	.198	32.24	52321.3	.CO1074

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFF. (PSIA)
1	.5	80.3	1.10	293.5	.199E+02
2	90.1	67.8	1.27	291.4	.199E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 48129. AVERAGE WALL REYNOLDS 36951. AVERAGE FRICTION FACTOR .00554

RUN 90HP, DATE 10/08/75, GAS HE-AR, MOLECULAR WT. = 15.83
 PIN = 72.8 P, TOUT = 547.4 P, MASS FLOW RATE = 25.1 LB/HR, I = 142.7 AMPS, E = 3.52 VOLTS
 PR,IM = .415, GR/RESQ = .166E-01, MACH(2) = .204, MACH(16) = .340, T,SURR = 79.5 P

IC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSLEIT	QGAS BTU/HR/FT ²	2*
2	.1	109.0	1.248	35671.6	.073	230.82	116316.4	.JJ2330
3	1.2	125.0	1.472	56221.1	.105	113.84	114231.9	.JJ2253
4	2.1	148.0	1.472	55827.7	.031	101.37	122912.0	.JJ2457
5	4.1	168.0	1.491	54951.1	.023	46.42	104446.7	.JJ2469
6	9.1	199.0	1.523	53350.0	.022	76.37	105755.6	.JJ2483
7	15.4	225.0	1.525	50410.0	.022	71.00	125787.1	.JJ2483
8	24.5	253.0	1.508	47907.0	.025	56.62	125740.6	.JJ2493
9	32.4	269.0	1.475	45769.9	.026	65.88	126287.7	.JJ2493
10	40.8	285.0	1.442	43763.3	.029	56.62	126315.1	.JJ2494
11	49.7	288.0	1.398	42033.3	.030	55.55	126508.3	.JJ2497
12	59.9	289.0	1.384	40478.8	.034	50.44	126455.6	.JJ2496
13	70.3	286.0	1.351	39108.8	.036	49.99	126499.2	.JJ2497
14	81.1	283.0	1.312	37806.6	.036	49.99	126498.6	.JJ2497
15	91.3	221.0	1.478	36650.0	.042	51.70	126223.4	.JJ2497
16	99.1	228.0	1.422	35538.8	.066	51.33	123450.7	.JJ2437
17	99.0	796.6	1.246	34792.2	.437	65.98	90759.6	.JJ1792

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFF. (PSIA)
1	.5	88.7	1.19	67.6	.512E+02
2	90.4	71.5	1.42	516.7	.283E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 46161. AVERAGE WALL REYNOLDS 30486. AVERAGE FRICTION FACTOR .00549

RUN 93HF, DATE 10/13/75, GAS HE-AR, MOLECULAR WT. = 15.83
 T_{IN} = 72.3 F, T_{OUT} = 348.8 F, MASS FLOW RATE = 14.1 LB/HR, I = 80.0 AMPS, E = 4.535 VOLTS
 PR,IN = .418, GR/RESQ = .187E-03, MACH(2) = .107, MACH(16) = .134, T, SURR = 75.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HR FT ²	Q+
2	.1	120.3	1.091	31673.	.067	192.11	3186.16	.JJ1289
	1.2	178.1	1.193	15233.	.140	83.56	3186.16	.JJ1289
	2.1	209.2	1.246	11401.	.054	70.00	3186.16	.JJ1289
	4.1	236.7	1.282	1125.	.026	7.99	3186.16	.JJ1289
	8.1	257.7	1.313	1060.7	.024	6.12	3186.16	.JJ1289
	16.4	266.1	1.326	959.7	.024	4.88	3186.16	.JJ1289
	32.8	271.1	1.346	868.7	.025	4.71	3186.16	.JJ1289
	65.6	273.5	1.307	787.2	.026	4.13	3186.16	.JJ1289
	98.4	277.7	1.288	706.7	.028	4.22	3186.16	.JJ1289
	161.2	281.1	1.271	633.9	.032	4.73	3186.16	.JJ1289
	194.0	283.9	1.255	560.0	.033	4.92	3186.16	.JJ1289
	226.8	286.2	1.246	485.1	.037	4.11	3186.16	.JJ1289
	259.6	287.7	1.234	388.5	.042	4.90	3186.16	.JJ1289
	292.4	291.9	1.207	233.2	.073	4.63	3186.16	.JJ1289
	325.2	291.6	1.078	229.12.	.273	97.21	3186.16	.JJ1289

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFER.
1	.5	95.9	1.07	71.5	-.589E-02
2	90.1	92.3	1.21	328.7	-.197E+01

AVERAGE BULK REYNOLDS 27508. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 22429. AVERAGE FRICTION FACTOR .00638

RUN 94HF, DATE 10/13/75, GAS HE-AR, MOLECULAR WT. = 15.83
 T_{IN} = 72.8 F, T_{OUT} = 348.2 F, MASS FLOW RATE = 14.1 LB/HR, I = 118.6 AMPS, E = 7.395 VOLTS
 PR,IN = .418, GR/RESQ = .407E-03, MACH(2) = .107, MACH(16) = .161, T, SURR = 73.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HR FT ²	Q+
2	.1	199.0	1.234	31584.	.093	158.74	79013.0	.JJ22775
	1.2	322.9	1.449	31259.	.148	77.42	75890.4	.JJ22775
	2.1	390.9	1.560	31001.	.053	66.89	83139.3	.JJ22775
	4.1	462.5	1.652	30411.	.037	56.25	84907.8	.JJ22775
	8.1	544.2	1.720	29347.	.036	47.47	85470.4	.JJ22775
	16.4	635.5	1.721	27407.	.037	41.08	85947.1	.JJ22775
	32.8	705.5	1.693	25803.	.041	37.37	85991.8	.JJ22775
	48.8	754.9	1.644	24462.	.043	35.69	86094.4	.JJ22775
	65.6	803.3	1.595	23203.	.048	34.25	85989.1	.JJ22775
	81.6	852.3	1.556	22169.	.052	33.00	85945.7	.JJ22775
	97.6	892.4	1.511	21237.	.057	32.47	85817.6	.JJ22775
	113.6	928.8	1.464	20417.	.059	32.62	85795.5	.JJ22775
	129.6	969.3	1.430	19640.	.066	32.01	85537.3	.JJ22775
	145.6	1006.2	1.395	18924.	.074	31.88	85360.9	.JJ22775
	161.6	1015.7	1.335	18282.	.119	33.33	81739.7	.JJ22775
	177.6	831.7	1.130	17894.	.885	49.25	47896.7	.JJ16883

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFER.
1	.5	95.8	1.18	73.2	-.589E-02
2	90.4	90.4	1.33	545.4	.300E+01

AVERAGE BULK REYNOLDS 24956. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 17318. AVERAGE FRICTION FACTOR .00655

RUN 98HP, DATE 10/20/75, GAS HE-AR, MOLECULAR WT. = 15.83
 TIN = 73.2 F, TOUT = 570.4 F, MASS FLOW RATE = 30.8 LB/HR, I = 160.9 AMPS, E = 9.670 VOLTS
 PR,IN = .413, GR/RESQ = .213E-03, MACH(2) = .204, MACH(16) = .340, T,SURR = 79.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HRFT ²	Q*
2	.1	235.9	1.259	39554.	.058	235.58	150418.8	.002417
3	1.2	353.4	1.517	68910.	.092	135.09	147299.2	.003365
4	2.1	418.9	1.631	68401.	.029	115.83	157043.5	.003533
5	4.1	490.3	1.729	67285.	.022	97.85	158341.7	.003557
6	8.1	559.8	1.787	65254.	.020	85.50	160013.2	.003571
7	16.4	641.1	1.791	61530.	.020	75.28	160388.0	.003585
8	24.6	705.7	1.772	58370.	.022	66.94	161295.5	.003599
9	32.5	759.9	1.744	55688.	.024	60.44	161779.5	.003613
10	40.9	809.6	1.707	53173.	.026	55.35	161935.1	.003627
11	49.9	859.9	1.680	51003.	.029	51.27	161977.6	.003641
12	59.0	894.4	1.636	49083.	.030	48.48	161958.2	.003655
13	68.4	931.1	1.601	47376.	.032	46.27	161833.9	.003669
14	77.7	970.9	1.568	45798.	.035	44.27	161579.3	.003683
15	87.1	996.6	1.527	44323.	.039	42.60	161199.9	.003697
16	96.4	1002.0	1.465	42947.	.059	41.99	157784.5	.003711
17	98.4	861.7	1.282	41994.	.407	71.18	118411.6	.001902

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	109.4	1.20	68.2	-.488E-02
2	90.4	88.7	1.46	537.9	.275E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 56296. AVERAGE WALL REYNOLDS 36396. AVERAGE FRICTION FACTOR .00517

RUN 97HP, DATE 10/20/75, GAS HE-AR, MOLECULAR WT. = 15.83
 TIN = 72.3 F, TOUT = 298.2 F, MASS FLOW RATE = 35.3 LB/HR, I = 117.2 AMPS, E = 5.790 VOLTS
 PR,IN = .418, GR/RESQ = .336E-04, MACH(2) = .238, MACH(16) = .337, T,SURR = 75.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HRFT ²	Q*
2	.1	131.5	1.129	80349.	.049	333.86	79933.4	.001124
3	1.2	199.2	1.253	80006.	.078	156.22	78204.2	.001099
4	2.1	231.6	1.309	79735.	.024	132.51	82545.1	.001160
5	4.1	264.6	1.359	79124.	.017	112.68	83345.5	.001171
6	8.1	291.3	1.385	77995.	.013	102.45	83824.7	.001179
7	16.4	321.0	1.389	75803.	.012	95.65	84077.0	.001181
8	24.6	351.3	1.396	73808.	.013	89.06	84153.5	.001192
9	32.5	366.9	1.379	71996.	.014	88.74	84302.6	.001194
10	40.9	390.2	1.373	70209.	.013	85.66	84314.8	.001194
11	49.9	409.9	1.361	68614.	.014	84.61	84433.8	.001186
12	59.0	433.1	1.360	67089.	.016	81.05	84444.7	.001185
13	68.4	457.9	1.350	65693.	.017	79.45	84496.5	.001187
14	77.7	483.1	1.314	64312.	.017	79.78	84511.5	.001187
15	87.1	508.4	1.324	63048.	.020	78.99	84428.2	.001186
16	96.4	538.6	1.285	61787.	.031	84.33	83508.6	.001173
17	98.0	447.8	1.213	60798.	.159	105.05	74059.2	.001040

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	106.9	1.10	63.5	-.473E-02
2	90.1	98.5	1.29	272.3	.193E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 71091. AVERAGE WALL REYNOLDS 53547. AVERAGE FRICTION FACTOR .00491

RUN 1024, DATE 01/23/76, GAS HE-AR, MOLECULAR WT. = 29.70
 IN = 70.1, TOUT = 375.9, MASS FLOW RATE = 36.4 LB/HR, I = 127.8 AMPS, E = 7.670 VOLTS
 PR,IN = .486, CR/RESQ = .755E-03, HACH(2) = .173, HACH(16) = .258, I, SURR = 77.0 F

TC	X/D	IN (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT2	Q+
2	1.1	197.7	1.246	82089.	.074	403.77	93352.5	.0022347
3	1.2	353.3	1.526	81162.	.157	171.51	87626.3	.0022250
4	2.1	429.9	1.650	80694.	.052	149.61	96330.4	.0022454
5	4.1	497.7	1.747	79114.	.034	129.60	99071.8	.0022543
6	9.1	573.3	1.815	76799.	.033	111.67	99727.5	.0022594
7	16.4	663.3	1.832	72178.	.034	96.47	100240.4	.0022573
8	24.6	726.4	1.807	68290.	.037	88.45	100391.2	.0022577
9	32.5	776.3	1.770	64994.	.040	83.42	100454.8	.0022595
10	40.9	820.3	1.723	61915.	.043	79.88	100454.8	.0022573
11	49.9	864.9	1.687	59322.	.046	76.59	100466.7	.0022573
12	57.0	903.3	1.646	56954.	.050	74.42	100362.5	.0022577
13	65.0	931.1	1.599	54893.	.052	74.06	100379.9	.0022577
14	73.4	969.6	1.505	52922.	.057	72.35	100186.0	.0022572
15	81.5	993.9	1.520	51135.	.062	72.78	99864.5	.0022564
16	90.4	999.8	1.458	49522.	.098	74.44	96564.4	.0022473
17	98.4	786.6	1.205	44437.	.568	114.98	66066.2	.0017172

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	110.6	1.20	97.0	-.470E-02
2	90.4	97.1	1.56	541.2	.245E+01

RUN 1038, DATE 01/23/76, GAS HE-AR, MOLECULAR WT. = 29.70
 IN = 71.0, TOUT = 471.8, MASS FLOW RATE = 36.2 LB/HR, I = 113.8 AMPS, E = 6.715 VOLTS
 PR,IN = .486, CR/RESQ = .593E-03, HACH(2) = .173, HACH(16) = .251, I, SURR = 90.0 F

TC	X/D	IN (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT2	Q+
2	1.1	191.6	1.234	31602.	.109	326.29	71673.1	.0018447
3	1.2	399.7	1.423	31002.	.146	169.15	69860.6	.0018500
4	2.1	354.5	1.521	30522.	.046	150.74	76889.8	.0019882
5	4.1	412.7	1.604	29421.	.035	127.99	77936.3	.0020100
6	9.1	462.8	1.684	27393.	.029	114.59	78785.5	.0020300
7	16.4	526.0	1.655	25587.	.029	101.97	79123.1	.0020399
8	24.6	574.6	1.642	22993.	.031	94.48	79231.0	.002042
9	32.5	610.7	1.614	67433.	.032	90.55	79364.3	.002045
10	40.9	649.4	1.589	64701.	.035	86.49	79300.6	.002044
11	49.9	682.5	1.561	62360.	.037	83.90	79300.6	.002045
12	56.8	715.0	1.534	60196.	.040	81.57	79301.2	.002044
13	64.9	740.6	1.502	58258.	.041	80.38	79336.5	.002045
14	73.3	767.8	1.472	56404.	.045	80.13	79237.5	.002042
15	81.5	793.0	1.443	54743.	.050	79.73	79005.6	.002036
16	90.3	799.1	1.393	51137.	.082	82.02	76660.3	.001976
17	98.2	634.9	1.175	20344.	.382	144.05	59365.5	.0015330

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	110.0	1.19	67.4	-.471E-02
2	90.3	98.3	1.39	444.1	.212E+01

AVERAGE BULK REYNOLDS 67411. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 4505. AVERAGE FRICTION FACTOR .00483

RUN 104H, DATE 01/28/76, GAS HE-AR, MOLECULAR WT. = 29.70
 VIN = 72.3 F, TOUT = 305.7 F, MASS FLOW RATE = 36.2 LB/HR, I = 87.4 AMPS, E = 5.020 VOLTS
 PR,IN = .486, GR/RESQ = .348E-03, MACH(2) = .173, MACH(16) = .223, T, SURR = 80.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HR/FT2	Q*
2	.1	149.3	1.153	81642.0	.119	299.35	4173.3	.001072
3	1.2	203.3	1.259	91275.0	.141	167.11	4173.3	.001036
4	2.1	237.7	1.309	30994.0	.042	151.83	45115.1	.001139
5	3.1	264.5	1.347	80344.0	.026	135.08	45339.1	.001130
6	4.1	293.3	1.377	79119.0	.023	121.02	46169.9	.001136
7	5.1	329.5	1.390	76716.0	.022	109.38	46304.3	.001130
8	6.5	355.5	1.397	74536.0	.023	104.34	46353.7	.001131
9	8.4	374.3	1.374	72577.0	.023	102.42	46423.2	.001193
10	9.8	393.9	1.368	70620.0	.026	98.20	46438.1	.001192
11	11.7	421.0	1.361	68880.0	.027	95.15	46416.0	.001192
12	13.6	439.6	1.348	67234.0	.028	93.75	46318.9	.001192
13	15.5	455.6	1.333	65708.0	.029	91.40	46417.5	.001192
14	17.3	475.9	1.323	64201.0	.032	91.35	46346.5	.001191
15	19.1	492.0	1.309	62827.0	.036	91.29	46244.0	.001198
16	21.0	496.1	1.278	61452.0	.060	95.16	45192.4	.001161
17	23.0	414.5	1.142	60411.0	.225	161.89	38861.3	.000998

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL
1	.5	110.1	1.12	88.0	-4.71E-02
2	90.1	101.1	1.28	288.4	.162E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 71562. AVERAGE WALL REYNOLDS 53128. AVERAGE FRICTION FACTOR .00484

RUN 105H, DATE 01/28/76, GAS HE-AR, MOLECULAR WT. = 29.70
 VIN = 71.9 F, TOUT = 197.5 F, MASS FLOW RATE = 36.3 LB/HR, I = 64.9 AMPS, E = 3.585 VOLTS
 PR,IN = .486, GR/RESQ = .193E-03, MACH(2) = .173, MACH(16) = .204, T, SURR = 79.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HR/FT2	Q*
2	.1	118.1	1.096	81892.0	.119	268.31	22971.1	.000590
3	1.2	147.5	1.149	81699.0	.136	164.83	22971.1	.000582
4	2.1	161.3	1.173	31541.0	.028	155.78	22979.0	.000648
5	3.1	177.1	1.176	81177.0	.025	139.57	25938.5	.000647
6	4.1	189.7	1.208	80493.0	.018	126.54	25942.8	.000692
7	5.1	189.7	1.208	79108.0	.018	117.60	25941.8	.000693
8	6.5	207.6	1.216	77799.0	.018	112.70	25942.8	.000693
9	8.4	221.9	1.215	75596.0	.019	110.75	25942.8	.000693
10	9.8	246.7	1.215	73357.0	.020	107.18	25941.9	.000693
11	11.7	257.9	1.212	74225.0	.021	105.46	25942.2	.000693
12	13.6	269.2	1.209	73129.0	.022	103.81	25941.1	.000693
13	15.5	278.2	1.202	72086.0	.023	104.13	25942.1	.000693
14	17.3	290.3	1.200	71032.0	.025	101.97	25938.5	.000692
15	19.1	298.9	1.183	70048.0	.028	102.70	25938.5	.000693
16	21.0	300.7	1.175	69037.0	.046	108.99	24336.6	.000693
17	23.0	262.1	1.098	68229.0	.145	182.22	22286.8	.000582

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL
1	.5	110.5	1.08	87.2	-4.71E-02
2	90.0	103.3	1.17	187.7	.130E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 75477. AVERAGE WALL REYNOLDS 61625. AVERAGE FRICTION FACTOR .00480

RUN 107H, DATE 01/28/76, GAS HE-AR, MOLECULAR WT. = 29.70
 T_{IN} = 72.3 F, T_{OUT} = 709.6 F, MASS FLOW RATE = 13.9 LB/HR, I = 90.0 AMPS, E = 5.370 VOLTS
 PR, IN = .486, GR/RESQ = .153E-02, MACH(2) = .363, MACH(16) = .123, T_{SURR} = 79.5 F

IC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR ²	Q*
2	.1	140.6	1.216	31054.	.115	209.01	44507.7	.002976
3	1.2	326.7	1.454	30713.	.273	86.007	45572.7	.003261
4	2.1	411.0	1.594	30439.	.418	74.000	39414.0	.003614
5	4.1	498.1	1.711	29799.	.077	61.000	47767.0	.003182
6	8.1	594.6	1.792	28443.	.073	51.000	47659.0	.003182
7	16.4	694.5	1.795	26570.	.078	41.000	47980.0	.003195
8	24.5	763.8	1.753	24880.	.084	40.000	47739.0	.003187
9	32.8	816.7	1.699	23484.	.090	37.000	47540.0	.003180
10	40.9	862.4	1.636	22208.	.099	36.000	47425.0	.003185
11	48.9	903.7	1.563	21153.	.105	35.000	47282.0	.003157
12	57.0	938.7	1.484	20215.	.114	34.000	47010.0	.003139
13	65.0	968.6	1.408	19396.	.122	34.000	46842.0	.003125
14	73.3	993.0	1.333	18621.	.131	34.000	46556.0	.003107
15	81.6	1013.2	1.259	17947.	.146	35.000	46017.0	.003072
16	90.4	1029.3	1.181	17328.	.161	35.000	45544.0	.003043
17	98.4	1043.5	1.102	17003.	.176	35.000	45077.0	.003014

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	89.1	1.18	74.0	-.591E-02
2	90.4	86.1	1.33	674.0	.297E+01

AVERAGE BULK REYNOLDS 24213. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 16879. AVERAGE FRICTION FACTOR .00642

RUN 108H, DATE 01/28/76, GAS HE-AR, MOLECULAR WT. = 29.70
 T_{IN} = 73.2 F, T_{OUT} = 519.7 F, MASS FLOW RATE = 14.0 LB/HR, I = 75.1 AMPS, E = 4.320 VOLTS
 PR, IN = .486, GR/RESQ = .106E-02, MACH(2) = .083, MACH(16) = .113, T_{SURR} = 80.0 F

IC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR ²	Q*
2	.1	148.6	1.177	31204.	.190	168.79	29250.3	.001945
3	1.2	254.2	1.326	30974.	.255	84.41	27664.7	.002182
4	2.1	306.1	1.413	30783.	.402	75.72	31682.7	.002185
5	4.1	360.4	1.487	30329.	.066	64.54	32956.8	.002218
6	8.1	414.5	1.534	29489.	.056	56.27	33230.0	.002212
7	16.4	430.8	1.544	27915.	.058	49.23	33370.3	.002219
8	24.5	526.7	1.524	26566.	.062	45.93	33377.5	.002218
9	32.8	566.0	1.499	25407.	.066	43.74	33345.0	.002216
10	40.9	599.5	1.464	24312.	.072	42.58	33272.4	.002210
11	48.9	639.5	1.439	23377.	.076	41.14	33176.1	.002205
12	57.0	662.7	1.405	22520.	.082	40.90	33017.3	.002198
13	65.0	696.9	1.376	21758.	.086	40.67	33003.9	.002194
14	73.3	722.3	1.352	21030.	.096	39.93	32776.7	.002179
15	81.6	748.4	1.321	20384.	.105	40.89	32559.9	.002164
16	90.4	775.7	1.279	19770.	.111	40.17	32233.4	.002140
17	98.4	801.6	1.033	19389.	.125	40.12	20186.9	.002142

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	88.9	1.14	73.6	-.333E-02
2	90.2	86.5	1.28	491.6	.233E+01

AVERAGE BULK REYNOLDS 25502. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 16693. AVERAGE FRICTION FACTOR .00627

RUN 109H, DATE 01/29/76, GAS HE-AR, MOLECULAR WT. = 29.70
 PIN = 74.1 F, FOUT = 361.3 F, MASS FLOW RATE = 13.9 LB/HR, I = 60.0 AMPS, E = 3.303 VOLTS
 PR,IN = .486, GR/RESQ = .681E-03, MACH(2) = .083, MACH(16) = .103, T, SUBR = 80.7 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR/FT2	Q*
2	.1	119.2	1.121	31140.	.197	154.86	18371.4	.001222
3	1.2	190.3	1.211	30995.	.256	83.98	17579.7	.001159
4	2.1	220.3	1.262	30872.	.090	77.30	20293.9	.001349
5	4.1	255.5	1.305	30578.	.053	57.58	20771.1	.001401
6	8.1	306.6	1.340	30020.	.047	58.22	18225.7	.001411
7	16.4	330.3	1.348	29945.	.046	55.52	17424.4	.001417
8	24.5	337.7	1.333	29831.	.047	55.52	17424.4	.001417
9	32.5	339.7	1.309	27121.	.051	49.99	16292.0	.001418
10	40.7	339.7	1.309	26276.	.054	48.04	15704.1	.001414
11	48.7	323.3	1.233	25534.	.062	46.67	14733.7	.001407
12	56.7	304.1	1.237	23577.	.071	46.67	13391.3	.001400
13	64.7	282.2	1.237	23005.	.080	46.67	12059.7	.001390
14	72.7	262.0	1.203	22452.	.145	46.67	11330.9	.001312
15	80.7	234.0	1.028	22064.	.429	29.26	15665.4	.001042

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL
1	.5	89.1	1.10	73.9	-.591E-02
2	90.1	87.1	1.20	341.5	.191E+01

AVERAGE BULK REYNOLDS 26805. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 21228. AVERAGE FRICTION FACTOR .30612

RUN 111H, DATE 01/29/76, GAS HE-AR, MOLECULAR WT. = 29.70
 PIN = 67.0 F, FOUT = 247.1 F, MASS FLOW RATE = 14.2 LB/HR, I = 48.0 AMPS, E = 2.923 VOLTS
 PR,IN = .486, GR/RESQ = .446E-03, MACH(2) = .084, MACH(16) = .097, T, SUBR = 72.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR/FT2	Q*
2	.1	101.7	1.067	32094.	.116	201.07	12579.6	.000833
3	1.2	139.0	1.134	31993.	.269	86.36	11097.4	.000734
4	2.1	160.0	1.170	31911.	.095	78.29	12874.2	.000852
5	4.1	179.7	1.199	31718.	.048	69.21	13474.4	.000892
6	8.1	201.9	1.223	31345.	.048	60.50	13376.4	.000899
7	16.4	227.2	1.234	30601.	.041	54.85	13607.2	.000901
8	24.5	245.2	1.233	29918.	.042	52.62	13599.5	.000901
9	32.5	251.4	1.229	29286.	.045	51.03	13369.5	.000900
10	40.7	257.3	1.223	28656.	.048	50.07	13369.5	.000900
11	48.7	232.1	1.213	28088.	.050	48.99	13369.5	.000900
12	56.7	207.0	1.210	27036.	.054	48.85	13369.5	.000900
13	64.7	199.3	1.203	27036.	.071	48.85	13369.5	.000900
14	72.7	174.4	1.194	25979.	.076	48.85	13369.5	.000900
15	80.7	156.0	1.191	25577.	.130	48.85	13369.5	.000900
16	88.7	139.3	1.191	25577.	.296	28.00	14298.0	.000934
17	96.7	122.0	1.191	25577.	.429	28.00	14298.0	.000934

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL
1	.5	89.4	1.03	66.5	-.587E-02
2	90.0	87.7	1.17	234.1	.158E+01

AVERAGE BULK REYNOLDS 29839. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 24231. AVERAGE FRICTION FACTOR .30593

RUN 112H, DATE 01/29/76, GAS HE-AR, MOLECULAR WT. = 29.70
 TIN = 67.9 K, TOUT = 132.9 F, MASS FLOW RATE = 14.2 LB/HR, I = 38.4 AMPS, E = 2.250 VOLTS
 PR,IN = .486, GR/RESQ = .284E-03, MACH(2) = .084, MACH(16) = .093, T, SURR = 72.3 F

TC	X/D	TW	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT2	Q*
2	.1	95.6	1.054	32001.	.172	151.80	7658.9	.000507
3	1.2	116.4	1.091	31938.	.227	84.67	7331.5	.000546
4	2.1	129.6	1.114	31884.	.087	75.75	8283.5	.000544
5	4.1	142.5	1.133	31759.	.050	66.48	8533.1	.000568
6	6.1	153.9	1.144	31520.	.036	61.16	8705.5	.000577
7	16.4	170.7	1.154	31032.	.038	55.38	8702.2	.000576
8	24.5	182.5	1.155	30571.	.039	53.28	8699.6	.000576
9	32.3	192.1	1.153	30141.	.040	52.53	8693.3	.000576
10	40.7	202.5	1.151	29702.	.044	51.48	8670.0	.000574
11	49.7	211.7	1.147	29296.	.044	51.12	8670.0	.000574
12	59.3	222.9	1.143	28900.	.049	49.42	8636.6	.000572
13	64.7	230.0	1.142	28528.	.050	49.90	8633.2	.000573
14	73.1	240.0	1.139	28177.	.054	49.45	8603.6	.000573
15	81.1	249.9	1.137	27790.	.064	48.35	8550.4	.000569
16	89.9	255.3	1.135	27432.	.116	49.35	8516.2	.000569
17	97.9	260.1	1.027	27149.	.234	217.49	7332.2	.000568

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	89.3	1.04	87.1	-.347E-02
2	90.0	87.9	1.12	174.4	.140E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 29720. AVERAGE WALL REYNOLDS 25498. AVERAGE FRICTION FACTOR .00590

RUN 116H, DATE 02/21/76, GAS HE-AR, MOLECULAR WT. = 27.53
 TIN = 70.1 K, TOUT = 154.2 F, MASS FLOW RATE = 44.1 LB/HR, I = 65.2 AMPS, E = 3.75 VOLTS
 PR,IN = .465, GR/RESQ = .808E-04, MACH(2) = .247, MACH(16) = .307, T, SURR = 73.3 F

TC	X/D	TW	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT2	Q*
2	.1	136.0	1.069	100497.	.057	430.33	24488.4	.000481
3	1.2	128.4	1.129	100309.	.130	188.90	24369.8	.000491
4	2.1	140.6	1.151	100162.	.027	174.92	24297.7	.000497
5	4.1	152.2	1.169	99822.	.018	154.26	24230.3	.000501
6	6.1	163.3	1.181	99179.	.014	142.00	24158.4	.000506
7	15.6	178.0	1.190	97897.	.014	131.66	24089.4	.000505
8	23.5	187.6	1.190	96690.	.014	128.76	24010.6	.000505
9	33.3	197.6	1.192	95548.	.015	124.98	23970.5	.000505
10	40.7	204.5	1.187	94370.	.015	122.98	23971.4	.000505
11	49.7	214.2	1.188	93303.	.016	122.87	23971.4	.000505
12	59.3	223.7	1.188	92249.	.017	120.28	23971.4	.000505
13	64.7	230.9	1.184	91243.	.017	120.46	23970.7	.000505
14	73.1	239.5	1.183	90219.	.018	119.31	23969.1	.000505
15	81.1	246.7	1.180	89269.	.021	119.52	23964.2	.000504
16	89.9	248.5	1.157	88285.	.035	126.47	23904.6	.000497
17	97.9	215.2	1.099	87476.	.096	225.52	23824.8	.000468

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	97.1	1.06	60.0	-.450E-02
2	90.0	84.2	1.17	147.3	.131E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 94402. AVERAGE WALL REYNOLDS 78875. AVERAGE FRICTION FACTOR .00464

RUN 117H, DATE 02/21/76, GAS HE-AR, MOLECULAR WT. = 27.53
 TIM = 70.1 F, TOUT = 287.8 F, MASS FLOW RATE = 44.2 LB/HR, I = 103.7 AMPS, E = 5.750 VOLTS
 PR,IN = .465, GR/RESQ = .195E-03, MACH(2) = .246, MACH(16) = .352, T,SURR = 74.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BIT/HR FT2	Q*
2	.1	131.7	1.136	100521.	.065	451.90	818156.6	.0011143
J	1.2	202.9	1.467	100069.	.114	206.43	35920.5	.0010996
5	2.1	235.7	1.324	99712.	.037	179.63	60203.2	.0011993
6	4.1	262.7	1.363	98903.	.020	160.09	61333.4	.0012022
7	9.1	292.1	1.393	97474.	.017	143.53	61607.4	.0012089
8	15.4	331.4	1.413	94475.	.018	128.26	61761.0	.0012111
9	24.5	356.1	1.409	91846.	.018	122.80	61965.7	.0012123
10	32.4	379.0	1.403	89457.	.019	118.39	61922.7	.0012114
11	40.8	398.6	1.389	87095.	.020	116.34	61944.0	.0012114
12	48.7	422.2	1.385	85011.	.021	112.17	61992.2	.0012115
13	56.8	439.1	1.371	83026.	.022	111.32	61991.4	.0012115
14	64.8	454.0	1.355	81207.	.022	111.68	62031.4	.0012115
15	73.1	473.8	1.346	79419.	.024	109.87	61931.5	.0012115
16	81.3	490.4	1.334	77779.	.027	109.50	61885.8	.0012113
17	90.1	491.8	1.300	76162.	.045	116.51	60861.7	.0011933
	98.0	415.6	1.170	74930.	.175	195.34	53799.3	.0010555

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	97.8	1.11	80.7	-.450E-02
2	90.1	80.5	1.30	272.1	.177E+01

AVERAGE BULK REYNOLDS 88371. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE HALL REYNOLDS 65677. AVERAGE FRICTION FACTOR .00453

RUN 118H, DATE 02/21/76, GAS HE-AR, MOLECULAR WT. = 27.53
 TIM = 70.1 F, TOUT = 407.9 F, MASS FLOW RATE = 44.9 LB/HR, I = 123.4 AMPS, E = 7.220 VOLTS
 PR,IN = .465, GR/RESQ = .354E-03, MACH(2) = .223, MACH(16) = .339, T,SURR = 78.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BIT/HR FT2	Q*
2	.1	179.3	1.221	101647.	.084	382.46	86114.1	.001662
J	1.2	278.0	1.400	100972.	.111	200.05	84677.3	.0015354
4	2.1	327.6	1.486	100431.	.036	174.60	1095.0	.0017388
5	4.1	375.1	1.554	99234.	.025	151.19	92425.0	.0017886
6	9.1	420.1	1.594	97016.	.021	135.21	93059.2	.0018394
7	16.4	475.1	1.605	92800.	.021	121.31	93448.2	.0018394
8	24.5	519.6	1.601	89109.	.022	112.57	93605.2	.0018077
9	32.4	554.4	1.584	85870.	.022	107.43	93754.4	.0018133
10	40.8	585.9	1.560	82755.	.023	103.88	93771.4	.0018133
11	48.7	613.3	1.535	80063.	.028	101.56	93909.9	.0018133
12	56.8	643.3	1.515	77558.	.028	98.75	93879.9	.0018133
13	64.8	670.2	1.494	75304.	.029	97.03	93934.6	.0018133
14	73.3	694.4	1.469	73127.	.032	96.28	93886.6	.0018133
15	81.4	713.5	1.441	71175.	.034	96.96	93763.3	.0018133
16	90.2	721.4	1.399	69273.	.057	99.87	91823.8	.0017722
17	98.2	605.4	1.227	67918.	.283	147.20	75005.9	.0014448

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	109.6	1.18	63.0	-.448E-02
2	90.2	90.6	1.40	384.6	.203E+01

AVERAGE BULK REYNOLDS 85507. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE HALL REYNOLDS 57344. AVERAGE FRICTION FACTOR .00456

RUN 119H, DATE 02/21/76, GAS HE-AR, MOLECULAR WT. = 27.53
 TIN = 71.9 F, TOUT = 452.4 F, MASS FLOW RATE = 44.7 LB/HR, I = 130.3 AMPS, E = 7.720 VOLTS
 PR,IN = .465, GR/RESQ = .392E-03, MACH(2) = .224, MACH(16) = .355, T,SURR = 91.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR/FT2	Q+
2	.1	195.0	1.247	101063.	.088	379.79	96546.3	.001862
3	1.2	310.6	1.455	100308.	.116	194.69	94971.1	.001833
4	2.1	360.7	1.540	99719.	.039	175.36	102933.1	.001895
5	4.1	411.2	1.609	98383.	.024	152.65	104166.6	.002003
6	8.1	463.1	1.655	95949.	.024	135.40	104820.0	.002003
7	16.4	533.1	1.675	91249.	.024	119.03	105273.1	.002033
8	24.5	584.8	1.671	87303.	.024	109.16	105467.7	.002033
9	32.4	624.4	1.650	83833.	.023	103.80	105688.3	.002033
10	40.7	657.7	1.616	80532.	.023	100.70	105689.1	.002033
11	48.8	690.0	1.590	77639.	.028	97.66	105821.9	.002044
12	56.9	723.1	1.565	75087.	.031	94.94	105796.6	.002043
13	64.7	752.2	1.538	72748.	.032	93.43	105865.9	.002042
14	73.1	781.1	1.514	70509.	.035	91.64	105789.9	.002043
15	81.2	807.7	1.485	68592.	.039	91.62	105602.2	.002037
16	90.3	807.3	1.429	66532.	.061	96.25	103362.8	.001933
17	98.2	675.3	1.243	65258.	.329	137.89	81768.2	.001577

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	109.1	1.19	64.8	-449E-02
2	90.3	88.5	1.43	427.0	.227E+01

AVERAGE BULK REYNOLDS 83880. AVERAGE WALL REYNOLDS 54686. AVERAGE FRICTION FACTOR .00460

RUN 120H, DATE 02/21/76, GAS HE-AR, MOLECULAR WT. = 27.53
 TIN = 71.9 F, TOUT = 208.2 F, MASS FLOW RATE = 43.7 LB/HR, I = 80.8 AMPS, E = 4.750 VOLTS
 PR,IN = .465, GR/RESQ = .121E-03, MACH(2) = .249, MACH(16) = .330, T,SURR = 80.7 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR/FT2	Q+
2	.1	131.0	1.132	99419.	.103	299.31	36159.9	.000713
3	1.2	165.5	1.195	99143.	.096	188.89	36484.6	.000722
4	2.1	185.4	1.230	98914.	.031	167.61	38836.4	.000756
5	4.1	204.1	1.258	98394.	.021	148.35	39274.5	.000775
6	8.1	217.9	1.269	97431.	.014	140.04	39551.2	.000780
7	16.4	242.7	1.285	95506.	.015	127.00	39501.1	.000781
8	24.5	262.5	1.292	93706.	.016	119.40	39614.7	.000781
9	32.4	272.0	1.281	92068.	.017	120.33	39678.3	.000783
10	40.7	289.6	1.283	90392.	.017	115.16	39645.0	.000782
11	48.8	302.7	1.279	88882.	.017	113.39	39691.8	.000783
12	56.9	316.8	1.276	87421.	.019	110.87	39661.3	.000782
13	64.7	327.3	1.269	86044.	.019	111.05	39634.8	.000783
14	73.1	341.4	1.266	84687.	.021	108.85	39648.9	.000782
15	81.2	352.0	1.259	83404.	.024	109.06	39585.8	.000781
16	90.3	353.9	1.237	82119.	.038	115.36	39029.6	.000773
17	97.9	316.2	1.161	81098.	.146	161.25	35272.3	.000696

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	95.3	1.11	61.7	-451E-02
2	90.0	80.6	1.24	197.8	.151E+01

AVERAGE BULK REYNOLDS 90788. AVERAGE WALL REYNOLDS 69459. AVERAGE FRICTION FACTOR .00464

RUN 124H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.33
 TIN = 70.5 F, TOUT = 183.2 F, MASS FLOW RATE = 14.2 LB/HR, I = 52.7 AMPS, E = 3.370 VOLTS
 PR,IN = .419, GR/RESQ = .682E-04, MACH(2) = .113, MACH(16) = .127, T,SURR = 74.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QJAS BTU/HRFT ²	Q*
2	.1	91.7	1.043	32217.	.057	183.33	15390.7	.0005540
3	1.2	116.6	1.088	32152.	.145	73.76	14790.6	.0005500
4	2.1	129.4	1.108	32102.	.040	70.71	16309.8	.0005531
5	4.1	140.0	1.125	31984.	.023	61.57	16583.2	.0005560
6	8.1	151.9	1.137	31758.	.018	55.39	16876.9	.0005563
7	16.4	167.2	1.145	31301.	.019	50.77	16893.4	.0005564
8	32.5	179.2	1.146	30867.	.019	48.72	16634.2	.0005564
9	64.3	189.2	1.145	30465.	.020	47.73	16691.6	.0005564
10	128.7	199.6	1.144	30051.	.022	46.80	16676.9	.0005563
11	256.9	209.6	1.143	29668.	.023	45.92	16675.6	.0005563
12	513.8	219.9	1.142	29294.	.025	44.97	16658.2	.0005563
13	1027.7	228.9	1.139	28939.	.026	44.68	16650.8	.0005563
14	2055.4	239.2	1.137	28576.	.028	43.84	16625.3	.0005562
15	4110.8	247.8	1.134	28236.	.032	43.78	16571.5	.0005563
16	8221.6	251.9	1.122	27888.	.055	45.84	16219.9	.0005548
17	16443.2	219.0	1.056	27604.	.148	93.90	14866.6	.0005512

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL (PSI)
1	.5	92.7	1.03	68.8	-0.58E-02
2	90.0	90.0	1.12	174.5	.140E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 30057. AVERAGE WALL REYNOLDS 26548. AVERAGE FRICTION FACTOR .00584

RUN 125H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.33
 TIN = 71.0 F, TOUT = 345.8 F, MASS FLOW RATE = 14.2 LB/HR, I = 51.5 AMPS, E = 4.305 VOLTS
 PR,IN = .419, GR/RESQ = .163E-03, MACH(2) = .113, MACH(16) = .142, T,SURR = 75.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QJAS BTU/HRFT ²	Q*
2	.1	126.4	1.106	32051.	.083	166.09	17434.2	.001267
3	1.2	182.3	1.205	31902.	.140	79.94	15721.2	.001209
4	2.1	210.9	1.253	31782.	.044	70.18	39079.5	.001323
5	4.1	239.1	1.292	31504.	.027	60.73	39823.1	.001348
6	8.1	269.3	1.320	30978.	.023	53.68	40054.8	.001356
7	16.4	307.3	1.332	29961.	.023	48.25	40154.4	.001359
8	32.5	335.2	1.327	29044.	.024	45.79	40192.7	.001360
9	64.3	358.5	1.316	28218.	.025	44.46	40223.7	.001361
10	128.7	382.3	1.305	27407.	.028	43.29	40192.0	.001360
11	256.9	405.4	1.295	26689.	.029	42.22	40217.4	.001361
12	513.8	428.4	1.285	26007.	.032	41.22	40165.1	.001359
13	1027.7	448.2	1.272	25381.	.033	40.94	40176.9	.001360
14	2055.4	470.7	1.261	24765.	.037	40.25	40131.7	.001357
15	4110.8	491.3	1.250	24203.	.041	39.84	39978.7	.001355
16	8221.6	502.1	1.235	23635.	.072	40.91	38357.9	.001315
17	16443.2	421.0	1.093	23210.	.275	81.51	32487.7	.001100

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFERENTIAL (PSI)
1	.5	92.7	1.08	89.9	-0.56E-02
2	90.1	89.0	1.22	325.8	.187E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 27853. AVERAGE WALL REYNOLDS 22239. AVERAGE FRICTION FACTOR .00599

RUN 126H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 71.9 P, TOUT = 167.5 P, MASS FLOW RATE = 24.5 LB/HR, I = 68.2 AMPS, E = 3.955 VOLTS
 PR,IN = .419, GR/RESQ = .273E-04, MACH(2) = .234, MACH(16) = .297, T,SURR = 76.3 P

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HRFT2	Q+
2	.1	133.4	1.077	30023.	.071	194.39	26469.8	.000518
3	1.2	136.6	1.119	30916.	.090	114.68	26052.6	.000509
4	2.1	138.0	1.139	31229.	.024	102.74	27754.6	.000543
5	4.1	148.1	1.153	32630.	.014	92.56	28163.2	.000551
7	8.1	159.6	1.165	34256.	.012	84.37	28128.8	.000550
8	15.4	174.6	1.174	35509.	.012	78.11	28161.9	.000551
9	24.5	186.3	1.176	33802.	.012	75.05	28172.9	.000551
10	32.4	195.9	1.175	33134.	.013	71.75	28180.6	.000551
11	40.7	205.9	1.174	32458.	.014	72.53	28176.2	.000551
12	48.7	216.1	1.174	31836.	.014	70.88	28181.5	.000551
13	56.8	226.0	1.174	31221.	.015	69.59	28173.0	.000551
14	64.7	234.6	1.171	30638.	.016	69.27	28174.7	.000551
15	73.0	244.0	1.170	30053.	.017	68.55	28150.4	.000551
16	81.2	252.6	1.167	29498.	.020	68.27	28108.0	.000553
17	89.9	254.8	1.153	28925.	.031	72.99	27790.9	.000543
17	97.9	234.7	1.107	28460.	.106	103.78	25883.9	.000556

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	76.7	1.06	82.8	-.514E-02
2	90.0	66.8	1.15	159.8	.142E+01

AVERAGE BULK REYNOLDS 52481. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 44157. AVERAGE FRICTION FACTOR .00504

RUN 127H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 73.2 P, TOUT = 510.4 P, MASS FLOW RATE = 14.1 LB/HR, I = 102.2 AMPS, E = 5.990 VOLTS
 PR,IN = .419, GR/RESQ = .256E-03, MACH(2) = .112, MACH(16) = .156, T,SURR = 78.5 P

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HRFT2	Q+
2	.1	152.7	1.149	31735.	.074	184.51	59526.6	.002016
3	1.2	246.1	1.313	31502.	.146	80.63	56131.8	.001901
4	2.1	290.2	1.385	31316.	.046	71.05	6137.6	.002091
5	4.1	334.9	1.443	30885.	.029	61.06	62919.0	.002131
7	8.1	383.3	1.484	30094.	.027	53.20	63302.9	.002144
8	16.4	444.9	1.492	28608.	.027	46.97	61534.3	.002152
9	24.5	489.5	1.476	27325.	.029	43.95	63620.6	.002155
10	32.4	525.6	1.451	26210.	.031	42.31	63695.6	.002158
11	40.7	560.4	1.423	25151.	.034	41.16	63651.5	.002156
12	48.7	594.0	1.399	24240.	.035	40.21	63739.0	.002159
13	56.8	613.7	1.391	23383.	.041	37.65	63552.0	.002153
14	64.7	610.4	1.367	22626.	.043	35.72	63530.9	.002154
15	73.0	611.5	1.344	21909.	.047	34.90	63463.9	.002153
16	81.2	701.0	1.321	21258.	.053	33.71	63232.2	.002142
17	89.9	744.3	1.291	20622.	.090	43.92	51124.7	.002138
17	98.2	613.6	1.106	20199.	.455	73.31	45405.6	.001538

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	92.7	1.12	72.8	-.588E-02
2	90.2	88.1	1.28	480.4	.236E+01

AVERAGE BULK REYNOLDS 26198. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 19810. AVERAGE FRICTION FACTOR .00605

RUN 128H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 74.1 F, TOUT = 688.6 F, MASS FLOW RATE = 14.1 LB/HR, I = 121.2 AMPS, Z = 7.280 VOLTS
 PR,IN = .419, GR/RESQ = .355E-03, MACH(2) = .113, MACH(16) = .171, T,SURR = 31.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HRFT2	Q+
2	.1	198.0	1.229	31682.	.082	163.34	83373.1	.002816
3	1.2	326.0	1.453	31357.	.145	77.77	79870.5	.002888
4	2.1	395.0	1.564	31095.	.051	67.10	87057.0	.002940
5	4.1	469.0	1.599	30499.	.036	56.17	88738.3	.002997
6	3.1	548.0	1.721	29427.	.014	47.83	89981.4	.003021
7	16.4	647.0	1.715	27484.	.036	40.66	89974.5	.003035
8	24.6	717.0	1.704	25876.	.040	37.13	89973.3	.003038
9	32.5	771.0	1.660	24527.	.043	35.15	90009.6	.003041
10	40.9	821.0	1.611	23266.	.048	33.63	89939.0	.003037
11	48.9	867.0	1.567	22229.	.051	32.63	89946.1	.003038
12	57.0	911.0	1.525	21293.	.056	31.84	89771.9	.003032
13	65.0	945.0	1.480	20470.	.060	31.78	89959.4	.003023
14	73.0	970.0	1.425	19694.	.061	32.82	89744.8	.003030
15	81.5	1026.0	1.408	18974.	.074	31.14	88959.9	.003034
16	90.4	1037.0	1.349	18330.	.117	32.38	85589.3	.002890
17	98.4	848.0	1.139	17941.	.275	46.57	50360.7	.001701

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	92.7	1.18	74.4	-.588E-02
2	90.4	87.0	1.35	650.4	.290E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 25029. AVERAGE WALL REYNOLDS 17357. AVERAGE FRICTION FACTOR .00616

RUN 129H, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 75.4 F, TOUT = 306.3 F, MASS FLOW RATE = 24.4 LB/HR, I = 102.4 AMPS, Z = 5.925 VOLTS
 PR,IN = .419, GR/RESQ = .609E-04, MACH(2) = .233, MACH(16) = .335, T,SURR = 82.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HRFT2	Q+
2	.1	145.2	1.148	55262.	.079	204.06	59450.1	.001162
3	1.2	202.0	1.249	55020.	.096	113.76	59758.9	.001189
4	2.1	229.0	1.295	54830.	.026	101.31	52852.8	.001229
5	4.1	256.0	1.333	54396.	.018	88.79	63512.7	.001242
6	3.1	284.0	1.361	53582.	.015	79.82	63791.5	.001247
7	16.4	321.0	1.376	51992.	.015	71.72	63959.8	.001251
8	24.6	349.0	1.375	50553.	.016	67.93	64045.7	.001252
9	32.5	371.0	1.367	49261.	.017	65.87	64119.4	.001254
10	40.9	394.0	1.358	47980.	.018	64.05	64191.1	.001254
11	48.9	416.0	1.350	46841.	.019	62.36	64126.6	.001254
12	57.0	437.0	1.342	45765.	.020	61.05	64191.1	.001254
13	65.0	456.0	1.331	44778.	.021	60.53	64179.8	.001254
14	73.0	475.0	1.321	43796.	.023	59.71	64223.2	.001254
15	81.5	495.0	1.311	42908.	.026	59.21	64189.6	.001254
16	90.4	500.0	1.281	42019.	.042	62.50	64093.8	.001254
17	98.0	447.0	1.183	41340.	.194	86.43	54878.2	.001073

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	76.7	1.12	67.0	-.515E-02
2	90.1	63.3	1.28	289.7	.194E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 48657. AVERAGE WALL REYNOLDS 36411. AVERAGE FRICTION FACTOR .00516

RUN 130R, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 76.3 F, TOUT = 451.5 F, MASS FLOW RATE = 24.3 LB/HR, I = 126.0 AMPS, E = 7.440 VOLTS
 PR,IN = .419, GR/RESQ = .143E-03, MACH(2) = .186, MACH(16) = .272, T,SURR = 34.7 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HRFT2	Q*
2	.1	176.9	1.196	45594.	.074	228.80	90625.7	.JJ1775
3	1.2	206.9	1.360	44332.	.100	113.99	39109.7	.JJ1747
4	2.1	226.9	1.432	43242.	.130	98.69	95469.5	.JJ1871
5	3.1	246.9	1.489	42281.	.170	84.12	96646.6	.JJ1894
6	4.1	266.9	1.530	41481.	.210	72.77	97152.2	.JJ1908
7	5.1	286.9	1.554	40778.	.250	62.50	97526.5	.JJ1912
8	6.1	306.9	1.564	40133.	.290	53.26	97719.9	.JJ1915
9	7.1	326.9	1.561	39527.	.330	45.00	97882.2	.JJ1913
10	8.1	346.9	1.544	38958.	.370	37.74	97913.3	.JJ1911
11	9.1	366.9	1.511	38425.	.410	31.18	98137.7	.JJ1908
12	10.1	386.9	1.455	37956.	.450	25.06	98055.4	.JJ1904
13	11.1	406.9	1.381	37595.	.490	19.31	97928.8	.JJ1898
14	12.1	426.9	1.338	37354.	.530	14.62	98081.4	.JJ1893
15	13.1	446.9	1.199	36201.	.570	10.10	97413.0	.JJ1517

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	96.3	1.15	71.9	-.516E-02
2	90.2	84.1	1.34	426.8	.222E+01

AVERAGE BULK REYNOLDS 45799. AVERAGE WALL REYNOLDS 32455. AVERAGE FRICTION FACTOR .JJ524

RUN 131R, DATE 03/01/76, GAS HE-AR, MOLECULAR WT. = 15.30
 TIN = 76.8 F, TOUT = 623.6 F, MASS FLOW RATE = 24.2 LB/HR, I = 151.1 AMPS, E = 9.100 VOLTS
 PR,IN = .419, GR/RESQ = .206E-03, MACH(2) = .186, MACH(16) = .305, T,SURR = 37.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK MUSSELT	QGAS BTU/HRFT2	Q*
2	.1	216.8	1.268	54345.	.067	228.80	131587.2	.JJ2582
3	1.2	354.1	1.509	53822.	.101	113.99	129787.5	.JJ2527
4	2.1	421.2	1.618	53404.	.133	98.69	138017.9	.JJ2709
5	3.1	489.8	1.707	52465.	.170	84.12	139860.7	.JJ2745
6	4.1	563.7	1.767	50784.	.210	72.77	146926.2	.JJ2754
7	5.1	658.1	1.783	47695.	.250	62.50	141568.9	.JJ2778
8	6.1	779.9	1.758	45111.	.290	53.26	141910.9	.JJ2785
9	7.1	929.3	1.722	42927.	.330	45.00	142163.9	.JJ2790
10	8.1	1099.3	1.678	40888.	.370	37.74	142237.3	.JJ2791
11	9.1	1287.3	1.640	39149.	.410	31.18	142391.6	.JJ2794
12	10.1	1491.1	1.601	37603.	.450	25.06	142387.7	.JJ2798
13	11.1	1711.1	1.556	36241.	.490	19.31	142450.1	.JJ2795
14	12.1	1956.1	1.519	34946.	.530	14.62	142406.5	.JJ2745
15	13.1	2224.1	1.482	33801.	.570	10.10	142263.1	.JJ2793
16	14.1	2516.1	1.421	32649.	.610	7.44	138833.6	.JJ2725
17	15.1	2831.1	1.237	31940.	.650	5.32	97328.7	.JJ1923

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	96.3	1.21	73.1	-.517E-02
2	90.4	80.9	1.42	588.0	.280E+01

AVERAGE BULK REYNOLDS 43534. AVERAGE WALL REYNOLDS 28483. AVERAGE FRICTION FACTOR .JJ526

RUN 133H, DATE 03/10/76, GAS HE, MOLECULAR WT. = 4.0026
 T_{IN} = 78.1 F, T_{OUT} = 174.0 F, MASS FLOW RATE = 11.6 LB/HR, I = 87.1 AMPS, E = 4.925 VOLTS
 PR, IN = .667, GR/RESQ = .112E-04, SACH(2) = .162, SACH(16) = .188, T_{SUBR} = 75.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT ²	Q*
2	.1	99	1.048	30117.	.040	240.47	44444.7	.000476
3	1.2	131	1.105	30065.	.072	95.70	43222.7	.000463
4	2.1	142	1.124	30026.	.114	45.18	45454.2	.000490
5	3.1	152	1.138	29976.	.162	25.53	45555.4	.000493
6	4.1	162	1.154	29926.	.210	15.98	46008.8	.000494
7	5.1	172	1.171	29876.	.258	11.53	46111.3	.000494
8	6.1	182	1.188	29826.	.306	8.08	46166.8	.000495
9	7.1	192	1.205	29776.	.354	5.63	46188.3	.000495
10	8.1	202	1.222	29726.	.402	4.18	46188.3	.000495
11	9.1	212	1.239	29676.	.450	3.73	46211.3	.000495
12	10.1	222	1.256	29626.	.498	3.28	46211.3	.000495
13	11.1	232	1.273	29576.	.546	2.83	46211.3	.000495
14	12.1	242	1.290	29526.	.594	2.38	46211.3	.000495
15	13.1	252	1.307	29476.	.642	1.93	46188.3	.000495
16	14.1	262	1.324	29426.	.690	1.48	45999.9	.000495
17	15.1	272	1.341	29376.	.738	1.03	45999.9	.000495
18	16.1	282	1.358	29326.	.786	.58	45999.9	.000495
19	17.1	292	1.375	29276.	.834	.13	45999.9	.000495
20	18.1	302	1.392	29226.	.882	.08	45999.9	.000495

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFECT
1	.5	103.1	1.04	73.9	-.536E-02
2	90.0	96.3	1.12	166.8	.151E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 28457. AVERAGE WALL REYNOLDS 25098. AVERAGE FRICTION FACTOR .00622

RUN 134H, DATE 03/10/76, GAS HE, MOLECULAR WT. = 4.0026
 T_{IN} = 77.7 F, T_{OUT} = 320.1 F, MASS FLOW RATE = 11.6 LB/HR, I = 135.4 AMPS, E = 7.390 VOLTS
 PR, IN = .667, GR/RESQ = .272E-04, SACH(2) = .162, SACH(16) = .211, T_{SUBR} = 78.0 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	QGAS BTU/HR FT ²	Q*
2	.1	142	1.123	30140.	.051	192.97	106727.1	.001143
3	1.2	211	1.250	30016.	.063	93.96	106024.4	.001135
4	2.1	238	1.395	29919.	.016	30.70	111919.7	.001190
5	3.1	263	1.328	29705.	.011	6.82	112277.1	.001130
6	4.1	289	1.351	29303.	.009	6.66	112277.1	.001130
7	5.1	322	1.360	28516.	.009	6.10	112277.1	.001130
8	6.1	346	1.354	27799.	.009	5.66	112277.1	.001130
9	7.1	366	1.343	27188.	.010	5.22	112277.1	.001130
10	8.1	386	1.328	26497.	.010	4.78	112277.1	.001130
11	9.1	407	1.321	25914.	.011	4.34	113309.3	.001131
12	10.1	427	1.310	25358.	.012	3.90	113319.7	.001131
13	11.1	444	1.297	24841.	.012	3.46	113326.4	.001131
14	12.1	464	1.287	24327.	.013	3.02	113329.5	.001131
15	13.1	481	1.274	23854.	.015	2.58	113326.4	.001131
16	14.1	483	1.240	23340.	.023	2.14	112379.0	.001131
17	15.1	452	1.169	22896.	.119	74.49	102462.4	.001097

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DIFFECT
1	.5	103.3	1.10	74.1	-.536E-02
2	90.1	94.2	1.24	301.4	.199E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 26749. AVERAGE WALL REYNOLDS 20976. AVERAGE FRICTION FACTOR .00634

RUN 135H, DATE 03/10/76, GAS HE, MOLECULAR WT. = 4.0026
 T_{IN} = 77.7 F, T_{OUT} = 471.7 F, MASS FLOW RATE = 11.6 LB/HR, I = 171.2 AMPS, E = 10.170 VOLTS
 PR, IN = .667, GR/RESQ = .435E-04, MACH(2) = .163, MACH(16) = .235, T, SURB = 32.0 F

TC	X/D	TW	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	HT/HT	Q*
2	.1	190.3	1.215	300091.	.065	179.547	21338.9	.001827
3	1.2	290.3	1.398	298994.	.061	178.841	17096.6	.001830
4	2.1	338.5	1.471	297411.	.057	178.135	17096.6	.001915
5	4.1	379.5	1.523	294023.	.053	177.429	18037.7	.001930
6	8.1	423.5	1.557	287773.	.051	176.723	18174.2	.001938
7	16.4	479.8	1.563	275886.	.049	176.017	18217.4	.001946
8	24.6	520.2	1.545	265336.	.047	175.311	18256.5	.001950
9	32.5	552.2	1.518	256166.	.045	174.605	18295.6	.001954
10	40.3	583.9	1.489	247255.	.043	173.899	18334.7	.001958
11	48.3	616.7	1.467	239499.	.041	173.193	18373.8	.001962
12	57.0	650.3	1.443	232999.	.039	172.487	18412.9	.001966
13	65.0	684.8	1.418	226725.	.037	171.781	18452.0	.001970
14	73.4	720.3	1.396	220684.	.035	171.075	18491.1	.001974
15	81.6	757.7	1.371	214861.	.033	170.369	18530.2	.001978
16	90.4	797.2	1.342	209255.	.031	169.663	18569.3	.001982
17	99.4	838.5	1.308	203866.	.028	168.957	18608.4	.001986
			1.226	200033.	.028	168.251	18647.5	.001990

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	103.1	1.16	74.7	-.596E-02
2	90.3	91.6	1.32	442.3	.253E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 25428. AVERAGE WALL REYNOLDS 18139. AVERAGE FRICTION FACTOR .00652

RUN 136H, DATE 03/10/76, GAS HE, MOLECULAR WT. = 4.0026
 T_{IN} = 76.8 F, T_{OUT} = 618.7 F, MASS FLOW RATE = 11.6 LB/HR, I = 199.6 AMPS, E = 12.015 VOLTS
 PR, IN = .667, GR/RESQ = .598E-04, MACH(2) = .162, MACH(16) = .258, T, SURB = 86.5 F

TC	X/D	TW	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	HT/HT	Q*
2	.1	230.2	1.289	300555.	.060	179.547	21338.9	.002510
3	1.2	369.2	1.533	297866.	.056	178.841	21338.9	.002511
4	2.1	434.9	1.640	295786.	.053	178.135	22448.8	.002525
5	4.1	493.7	1.711	291177.	.049	177.429	22448.8	.002549
6	8.1	559.9	1.760	282811.	.047	176.723	22448.8	.002563
7	16.4	642.3	1.760	267355.	.045	176.017	22448.8	.002577
8	24.6	699.9	1.725	254277.	.043	175.311	22448.8	.002586
9	32.5	744.4	1.682	243033.	.041	174.605	22448.8	.002592
10	40.3	787.7	1.633	232011.	.039	173.899	22448.8	.002598
11	48.3	829.0	1.594	222655.	.037	173.193	22448.8	.002601
12	57.0	867.2	1.554	214600.	.035	172.487	22448.8	.002605
13	65.0	909.9	1.513	207337.	.033	171.781	22448.8	.002608
14	73.4	938.8	1.479	200442.	.031	171.075	22448.8	.002611
15	81.6	977.7	1.446	194199.	.029	170.369	22448.8	.002614
16	90.4	977.7	1.382	188413.	.028	169.663	22448.8	.002618
17	99.4	997.4	1.258	183688.	.026	168.957	20400.9	.002611

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PRESS DEFECT
1	.5	103.2	1.22	74.5	-.596E-02
2	90.4	83.2	1.38	579.3	.309E+01

AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE BULK REYNOLDS 24454. AVERAGE WALL REYNOLDS 16195. AVERAGE FRICTION FACTOR .00669

RUN 1494, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 74.5 F, TOUT = 153.5 F, MASS FLOW RATE = 35.6 LB/HR, I = 64.24MPS, E = 3.55 VOLTS
 PR. IN = .720, GR/RESQ = .162E-03, MACH(2) = .225, MACH(1) = .250, T. SURR = 75.7 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/GAS	BULK NUSSLEIT	OGAS BTU/HR FT2	Q+
1	.1	130.3	1.1264	100229.	.074	535.1	2107.4	.000417
1	.2	137.0	1.1266	100099.	.145	530.4	2333.7	.000432
1	.3	143.9	1.1268	99977.	.031	224.4	2447.4	.000436
1	.4	160.0	1.1264	99657.	.019	204.4	2247.7	.000444
1	.5	171.1	1.1276	99137.	.015	193.4	2247.4	.000443
1	.6	183.0	1.1289	98056.	.015	175.4	2246.4	.000444
1	.7	195.5	1.1288	97029.	.015	153.4	2246.4	.000444
1	.8	203.3	1.1286	96050.	.015	153.4	2246.4	.000445
1	.9	211.1	1.1283	95072.	.016	153.4	2246.4	.000444
1	1.0	220.0	1.1283	94151.	.016	153.4	2246.4	.000445
1	1.1	229.0	1.1283	93228.	.017	153.4	2246.4	.000444
1	1.2	238.0	1.1280	92319.	.018	153.4	2246.4	.000444
1	1.3	247.0	1.1278	91419.	.018	153.4	2246.4	.000444
1	1.4	256.0	1.1278	90522.	.022	153.4	2246.4	.000444
1	1.5	265.0	1.1278	89622.	.022	153.4	2246.4	.000444
1	1.6	274.0	1.1278	88722.	.022	153.4	2246.4	.000444
1	1.7	283.0	1.1278	87822.	.022	153.4	2246.4	.000444
1	1.8	292.0	1.1278	86922.	.022	153.4	2246.4	.000444
1	1.9	301.0	1.1278	86022.	.022	153.4	2246.4	.000444
1	2.0	310.0	1.1278	85122.	.022	153.4	2246.4	.000444
1	2.1	319.0	1.1278	84222.	.022	153.4	2246.4	.000444
1	2.2	328.0	1.1278	83322.	.022	153.4	2246.4	.000444
1	2.3	337.0	1.1278	82422.	.022	153.4	2246.4	.000444
1	2.4	346.0	1.1278	81522.	.022	153.4	2246.4	.000444
1	2.5	355.0	1.1278	80622.	.022	153.4	2246.4	.000444
1	2.6	364.0	1.1278	79722.	.022	153.4	2246.4	.000444
1	2.7	373.0	1.1278	78822.	.022	153.4	2246.4	.000444
1	2.8	382.0	1.1278	77922.	.022	153.4	2246.4	.000444
1	2.9	391.0	1.1278	77022.	.022	153.4	2246.4	.000444
1	3.0	400.0	1.1278	76122.	.022	153.4	2246.4	.000444
1	3.1	409.0	1.1278	75222.	.022	153.4	2246.4	.000444
1	3.2	418.0	1.1278	74322.	.022	153.4	2246.4	.000444
1	3.3	427.0	1.1278	73422.	.022	153.4	2246.4	.000444
1	3.4	436.0	1.1278	72522.	.022	153.4	2246.4	.000444
1	3.5	445.0	1.1278	71622.	.022	153.4	2246.4	.000444
1	3.6	454.0	1.1278	70722.	.022	153.4	2246.4	.000444
1	3.7	463.0	1.1278	69822.	.022	153.4	2246.4	.000444
1	3.8	472.0	1.1278	68922.	.022	153.4	2246.4	.000444
1	3.9	481.0	1.1278	68022.	.022	153.4	2246.4	.000444
1	4.0	490.0	1.1278	67122.	.022	153.4	2246.4	.000444
1	4.1	499.0	1.1278	66222.	.022	153.4	2246.4	.000444
1	4.2	508.0	1.1278	65322.	.022	153.4	2246.4	.000444
1	4.3	517.0	1.1278	64422.	.022	153.4	2246.4	.000444
1	4.4	526.0	1.1278	63522.	.022	153.4	2246.4	.000444
1	4.5	535.0	1.1278	62622.	.022	153.4	2246.4	.000444
1	4.6	544.0	1.1278	61722.	.022	153.4	2246.4	.000444
1	4.7	553.0	1.1278	60822.	.022	153.4	2246.4	.000444
1	4.8	562.0	1.1278	59922.	.022	153.4	2246.4	.000444
1	4.9	571.0	1.1278	59022.	.022	153.4	2246.4	.000444
1	5.0	580.0	1.1278	58122.	.022	153.4	2246.4	.000444
1	5.1	589.0	1.1278	57222.	.022	153.4	2246.4	.000444
1	5.2	598.0	1.1278	56322.	.022	153.4	2246.4	.000444
1	5.3	607.0	1.1278	55422.	.022	153.4	2246.4	.000444
1	5.4	616.0	1.1278	54522.	.022	153.4	2246.4	.000444
1	5.5	625.0	1.1278	53622.	.022	153.4	2246.4	.000444
1	5.6	634.0	1.1278	52722.	.022	153.4	2246.4	.000444
1	5.7	643.0	1.1278	51822.	.022	153.4	2246.4	.000444
1	5.8	652.0	1.1278	50922.	.022	153.4	2246.4	.000444
1	5.9	661.0	1.1278	50022.	.022	153.4	2246.4	.000444
1	6.0	670.0	1.1278	49122.	.022	153.4	2246.4	.000444
1	6.1	679.0	1.1278	48222.	.022	153.4	2246.4	.000444
1	6.2	688.0	1.1278	47322.	.022	153.4	2246.4	.000444
1	6.3	697.0	1.1278	46422.	.022	153.4	2246.4	.000444
1	6.4	706.0	1.1278	45522.	.022	153.4	2246.4	.000444
1	6.5	715.0	1.1278	44622.	.022	153.4	2246.4	.000444
1	6.6	724.0	1.1278	43722.	.022	153.4	2246.4	.000444
1	6.7	733.0	1.1278	42822.	.022	153.4	2246.4	.000444
1	6.8	742.0	1.1278	41922.	.022	153.4	2246.4	.000444
1	6.9	751.0	1.1278	41022.	.022	153.4	2246.4	.000444
1	7.0	760.0	1.1278	40122.	.022	153.4	2246.4	.000444
1	7.1	769.0	1.1278	39222.	.022	153.4	2246.4	.000444
1	7.2	778.0	1.1278	38322.	.022	153.4	2246.4	.000444
1	7.3	787.0	1.1278	37422.	.022	153.4	2246.4	.000444
1	7.4	796.0	1.1278	36522.	.022	153.4	2246.4	.000444
1	7.5	805.0	1.1278	35622.	.022	153.4	2246.4	.000444
1	7.6	814.0	1.1278	34722.	.022	153.4	2246.4	.000444
1	7.7	823.0	1.1278	33822.	.022	153.4	2246.4	.000444
1	7.8	832.0	1.1278	32922.	.022	153.4	2246.4	.000444
1	7.9	841.0	1.1278	32022.	.022	153.4	2246.4	.000444
1	8.0	850.0	1.1278	31122.	.022	153.4	2246.4	.000444
1	8.1	859.0	1.1278	30222.	.022	153.4	2246.4	.000444
1	8.2	868.0	1.1278	29322.	.022	153.4	2246.4	.000444
1	8.3	877.0	1.1278	28422.	.022	153.4	2246.4	.000444
1	8.4	886.0	1.1278	27522.	.022	153.4	2246.4	.000444
1	8.5	895.0	1.1278	26622.	.022	153.4	2246.4	.000444
1	8.6	904.0	1.1278	25722.	.022	153.4	2246.4	.000444
1	8.7	913.0	1.1278	24822.	.022	153.4	2246.4	.000444
1	8.8	922.0	1.1278	23922.	.022	153.4	2246.4	.000444
1	8.9	931.0	1.1278	23022.	.022	153.4	2246.4	.000444
1	9.0	940.0	1.1278	22122.	.022	153.4	2246.4	.000444
1	9.1	949.0	1.1278	21222.	.022	153.4	2246.4	.000444
1	9.2	958.0	1.1278	20322.	.022	153.4	2246.4	.000444
1	9.3	967.0	1.1278	19422.	.022	153.4	2246.4	.000444
1	9.4	976.0	1.1278	18522.	.022	153.4	2246.4	.000444
1	9.5	985.0	1.1278	17622.	.022	153.4	2246.4	.000444
1	9.6	994.0	1.1278	16722.	.022	153.4	2246.4	.000444
1	9.7	1003.0	1.1278	15822.	.022	153.4	2246.4	.000444
1	9.8	1012.0	1.1278	14922.	.022	153.4	2246.4	.000444
1	9.9	1021.0	1.1278	14022.	.022	153.4	2246.4	.000444
1	10.0	1030.0	1.1278	13122.	.022	153.4	2246.4	.000444
1	10.1	1039.0	1.1278	12222.	.022	153.4	2246.4	.000444
1	10.2	1048.0	1.1278	11322.	.022	153.4	2246.4	.000444
1	10.3	1057.0	1.1278	10422.	.022	153.4	2246.4	.000444
1	10.4	1066.0	1.1278	9522.	.022	153.4	2246.4	.000444
1	10.5	1075.0	1.1278	8622.	.022	153.4	2246.4	.000444
1	10.6	1084.0	1.1278	7722.	.022	153.4	2246.4	.000444
1	10.7	1093.0	1.1278	6822.	.022	153.4	2246.4	.000444
1	10.8	1102.0	1.1278	5922.	.022	153.4	2246.4	.000444
1	10.9	1111.0	1.1278	5022.	.022	153.4	2246.4	.000444
1	11.0	1120.0	1.1278	4122.	.022	153.4	2246.4	.000444
1	11.1	1129.0	1.1278	3222.	.022	153.4	2246.4	.000444
1	11.2	1138.0	1.1278	2322.	.022	153.4	2246.4	.000444
1	11.3	1147.0	1.1278	1422.	.022	153.4	2246.4	.000444
1	11.4	1156.0	1.1278	522.	.022	153.4	2246.4	.000444
1	11.5	1165.0	1.1278	0.	.022	153.4	2246.4	.000444

PT X/D STATIC PRESS. (PSIA) TW/TB TB (F) PRESS. DIFF. (PSI) FRICTION FACTOR
 1 =.5 92.1 1.05 66.0 =.490 F=02
 2 90.0 84.4 1.16 151.7 =.117 F=01

AVERAGE BULK REYNOLDS 94992. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 80621. AVERAGE FRICTION FACTOR .00451

RUN 1504, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 73.9 F, TOUT = 225.5 F, MASS FLOW RATE = 35.4 LB/HR, I = 69.14MPS, E = 5.720 VOLTS
 PR. IN = .720, GR/RESQ = .386E-03, MACH(2) = .224, MACH(1) = .293, T. SURR = 79.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/GAS	BULK NUSSLEIT	OGAS BTU/HR FT2	Q+
2	.1	151.5	1.150	39494.	.092	504.4	3501.1	.000943
1	.2	160.0	1.1203	33120.	.137	504.4	3501.1	.000943
1	.3	168.0	1.1203	28827.	.051	215.4	3501.1	.000943
1	.4	176.0	1.1203	24534.	.031	191.4	3501.1	.000943
1	.5	184.0	1.1203	20241.	.019	174.4	3501.1	.000943
1	.6	192.0	1.1203	15948.	.015	163.4	3501.1	.000943
1	.7	200.0	1.1203	11655.	.015	153.4	3501.1	.000943
1	.8	208.0	1.1203	7362.	.015	143.4	3501.1	.000943
1	.9	216.0	1.1203	3069.	.015	133.4	3501.1	.000943
1	1.0	224.0	1.1203	0.	.015	123.4	3501.1	.000943
1	1.1	232.0	1.1203	0.	.015	113.4	3501.1	.000943
1	1.2	240.0	1.1203	0.	.015	103.4	3501.1	.000943
1	1.3	248.0	1.1203	0.	.015	93.4	3501.1	.000943

RUN 1544, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 77.7 F, TOUT = 171.2 F, MASS FLOW RATE = 29.3 LB/HR, I = 35.5 A, PS, E = 3.260 VOLTS
 PR. IN = .719, GR/RESO = .239E-03, MACH(2) = .170, MACH(16) = .168, T. SUPR = 32.0 F

TC	X/D	T ₁ (F)	T _w /T _s	BULK REYNOLDS	HL/OGAS	BULK NUSSLELT	CGAS BTU/HRFT ²	Q+
2	.1	109.1	1.063	3110.4	.083	43.1	201.2	.000437
3	.2	141.4	1.123	3100.3	.108	43.1	201.2	.000437
4	.3	157.7	1.142	3099.2	.133	43.1	201.2	.000437
5	.4	170.0	1.174	3098.1	.158	43.1	201.2	.000437
6	.5	182.3	1.181	3097.0	.183	43.1	201.2	.000437
7	.6	190.1	1.132	3095.9	.208	43.1	201.2	.000437
8	.7	201.1	1.180	3094.8	.233	43.1	201.2	.000437
9	.8	213.3	1.132	3093.7	.258	43.1	201.2	.000437
10	.9	222.7	1.179	3092.6	.283	43.1	201.2	.000437
11	1.0	233.3	1.177	3091.5	.308	43.1	201.2	.000437
12	1.1	244.3	1.174	3090.4	.333	43.1	201.2	.000437
13	1.2	255.1	1.172	3089.3	.358	43.1	201.2	.000437
14	1.3	259.3	1.168	3088.2	.383	43.1	201.2	.000437
15	1.4	259.3	1.153	3087.1	.408	43.1	201.2	.000437
16	1.5	259.3	1.153	3086.0	.433	43.1	201.2	.000437
17	1.6	220.1	1.077	3084.9	.458	43.1	201.2	.000437

PT	X/D	STATIC PRESS. (PSIA)	T _w /T _s	T _B (F)	PRESS DIFFERENTIAL
1	.5	99.9	1.05	75.0	.47
2	90.0	95.2	1.15	163.8	.11

AVERAGE BULK REYNOLDS 76666. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 69365. AVERAGE FRICTION FACTOR .00459

RUN 1554, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 76.6 F, TOUT = 238.5 F, MASS FLOW RATE = 29.1 LB/HR, I = 36.6 A, PS, E = 5.220 VOLTS
 PR. IN = .719, GR/RESO = .544E-03, MACH(2) = .170, MACH(16) = .212, T. SUPR = 33.0 F

TC	X/D	T ₁ (F)	T _w /T _s	BULK REYNOLDS	HL/OGAS	BULK NUSSLELT	CGAS BTU/HRFT ²	Q+
2	.1	191.4	1.139	3106.5	.099	43.2	456.7	.000990
3	1.2	225.4	1.174	3076.1	.156	43.2	456.7	.000990
4	2.1	262.4	1.133	3051.0	.251	43.2	456.7	.000990
5	4.1	293.0	1.138	3024.0	.329	43.2	456.7	.000990
6	3.1	321.0	1.109	2993.4	.424	43.2	456.7	.000990
7	16.4	354.0	1.120	2964.1	.523	43.2	456.7	.000990
8	24.5	373.0	1.115	2935.3	.623	43.2	456.7	.000990
9	32.4	397.0	1.110	2906.7	.723	43.2	456.7	.000990
10	40.8	413.0	1.113	2878.3	.823	43.2	456.7	.000990
11	43.7	437.0	1.118	2850.0	.923	43.2	456.7	.000990
12	56.8	457.0	1.123	2821.6	1.023	43.2	456.7	.000990
13	64.8	471.0	1.128	2793.3	1.123	43.2	456.7	.000990
14	73.2	489.0	1.133	2765.0	1.223	43.2	456.7	.000990
15	81.1	503.0	1.138	2736.7	1.323	43.2	456.7	.000990
16	90.1	506.0	1.143	2708.4	1.423	43.2	456.7	.000990
17	98.0	403.0	1.140	2680.1	1.523	43.2	456.7	.000990

PT	X/D	STATIC PRESS. (PSIA)	T _w /T _s	T _B (F)	PRESS DIFFERENTIAL
1	.5	99.9	1.11	76.6	.47
2	90.1	93.7	1.30	262.0	.151

AVERAGE BULK REYNOLDS 72340. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 54980. AVERAGE FRICTION FACTOR .00475

RUN 1564, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 79.0 F, TOUT = 423.3 F, MASS FLOW RATE = 29.0 LB/HR, I = 12.3 AMP, E = 6.665 VOLTS
 PR, IN = .719, GR/RESO = .642E-03, MACH(2) = .170, MACH(10) = .231, T, SUPR = 92.5 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/CGAS	BULK NUSSLELT	CGAS BTU/HR FT ²	Q+
2	1	195.2	1.22	30768.	.108	413.56	76446.6	.001529
1	1	202.7	1.46	32297.	.161	200.53	47438.5	.001471
2	1	202.7	1.55	32297.	.050	193.37	75253.7	.001633
3	1	202.7	1.55	73074.	.033	134.21	76817.0	.001565
4	1	202.7	1.55	77522.	.029	114.15	77402.1	.001674
5	1	202.7	1.55	74557.	.023	127.21	77745.0	.001680
6	1	202.7	1.55	71532.	.029	113.15	77376.0	.001649
7	1	202.7	1.55	53534.	.030	114.15	75022.3	.001642
8	1	202.7	1.55	66723.	.033	109.49	77460.9	.001641
9	1	202.7	1.55	52313.	.033	108.14	75061.1	.001641
10	1	202.7	1.55	63438.	.036	102.50	77488.6	.001692
11	1	202.7	1.55	60177.	.037	101.60	75046.2	.001693
12	1	202.7	1.55	60313.	.040	99.49	77446.7	.001693
13	1	202.7	1.55	52313.	.044	98.44	77769.6	.001647
14	1	202.7	1.55	52313.	.073	102.33	75492.3	.001647
15	1	202.7	1.55	52313.	.096	153.16	51953.0	.001344

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TS (F)	PRESS DIFFERENTIAL (PSI)
1	90.5	90.7	1.17	78.3	-.47
2	90.5	92.1	1.40	89.4	-.14

AVERAGE BULK REYNOLDS 69032. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 46754. AVERAGE FRICTION FACTOR .00479

RUN 1574, DATE 04/02/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 79.5 F, TOUT = 553.5 F, MASS FLOW RATE = 28.0 LB/HR, I = 12.2 AMP, E = 7.960 VOLTS
 PR, IN = .719, GR/RESO = .116E-02, MACH(2) = .170, MACH(10) = .250, T, SUPR = 88.7 F

TC	X/D	TW (F)	TW/TB	BULK REYNOLDS	HL/CGAS	BULK NUSSLELT	CGAS BTU/HR FT ²	Q+
2	1	243.2	1.305	30324.	.109	409.61	37118.6	.002112
1	1	401.4	1.584	73677.	.141	200.53	33378.3	.002038
2	1	479.9	1.715	73162.	.051	179.40	134178.3	.002038
3	1	544.9	1.409	73001.	.037	156.20	105009.2	.002038
4	1	515.5	1.863	75053.	.033	135.27	107009.3	.002038
5	1	609.4	1.863	72030.	.033	121.73	10779.3	.002038
6	1	740.4	1.834	53674.	.036	112.23	10779.3	.002038
7	1	792.0	1.792	53466.	.037	107.22	10779.3	.002038
8	1	834.4	1.748	53119.	.041	101.60	10779.3	.002038
9	1	873.3	1.708	60337.	.043	97.47	10779.3	.002038
10	1	903.3	1.667	53171.	.046	95.21	10779.3	.002038
11	1	933.3	1.619	56859.	.047	94.22	10779.3	.002038
12	1	963.3	1.581	55036.	.051	92.90	10779.3	.002038
13	1	993.3	1.543	53130.	.056	91.44	10779.3	.002038
14	1	993.3	1.479	53188.	.088	93.43	10779.3	.002038
15	1	766.1	1.207	50863.	.443	153.92	77331.6	.001822

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TS (F)	PRESS DIFFERENTIAL (PSI)
1	90.4	90.6	1.24	78.3	-.47
2	90.4	90.6	1.43	88.8	-.22

AVERAGE BULK REYNOLDS 66150. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 41168. AVERAGE FRICTION FACTOR .00479

RUN 150H, DATE 04/05/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 73.7 F, TOUT = 175.5 F, MASS FLOW RATE = 20.1 LB/HR, I = 52.2 A, E = 2.33C VOLTS
 PR.IN = .72C, GR/RESS = .214E-03, MACH(2) = .157, MACH(10) = .174, T.SUR = 76.0 F

TC	X/D	T4 (F)	TW/TB	BULK REYNOLDS	HL/GAS	BULK NUSSLELT	CGAS BTU/HR FT2	Q+
1	.1	140.3	1.114	5555	.119	340.4	3420	.001083
2	.1	133.9	1.110	5643	.120	340.4	3194	.001011
3	.1	151.6	1.133	5588	.071	138.0	3610	.001161
4	.1	163.6	1.103	5546	.040	121.0	3721	.001177
5	.1	177.7	1.104	5467	.033	109.8	3762	.001189
6	.1	192.2	1.104	5315	.022	100.0	3775	.001193
7	.1	203.2	1.104	5175	.022	95.0	3780	.001194
8	.1	212.2	1.104	5047	.022	92.0	3780	.001194
9	.1	222.2	1.104	4913	.022	89.0	3780	.001194
10	.1	232.2	1.104	4780	.022	86.0	3780	.001194
11	.1	240.0	1.104	4648	.022	83.0	3780	.001194
12	.1	250.0	1.104	4513	.022	80.0	3780	.001194
13	.1	260.0	1.104	4379	.022	77.0	3780	.001194
14	.1	266.0	1.104	4244	.022	74.0	3780	.001194
15	.1	272.0	1.104	4109	.022	71.0	3780	.001194
16	.1	277.0	1.104	3974	.022	68.0	3780	.001194
17	.1	283.0	1.104	3839	.022	65.0	3780	.001194
18	.1	288.0	1.104	3704	.022	62.0	3780	.001194
19	.1	294.0	1.104	3569	.022	59.0	3780	.001194
20	.1	300.0	1.104	3434	.022	56.0	3780	.001194
21	.1	306.0	1.104	3299	.022	53.0	3780	.001194
22	.1	312.0	1.104	3164	.022	50.0	3780	.001194
23	.1	318.0	1.104	3029	.022	47.0	3780	.001194
24	.1	324.0	1.104	2894	.022	44.0	3780	.001194
25	.1	330.0	1.104	2759	.022	41.0	3780	.001194
26	.1	336.0	1.104	2624	.022	38.0	3780	.001194
27	.1	342.0	1.104	2489	.022	35.0	3780	.001194
28	.1	348.0	1.104	2354	.022	32.0	3780	.001194
29	.1	354.0	1.104	2219	.022	29.0	3780	.001194
30	.1	360.0	1.104	2084	.022	26.0	3780	.001194
31	.1	366.0	1.104	1949	.022	23.0	3780	.001194
32	.1	372.0	1.104	1814	.022	20.0	3780	.001194
33	.1	378.0	1.104	1679	.022	17.0	3780	.001194
34	.1	384.0	1.104	1544	.022	14.0	3780	.001194
35	.1	390.0	1.104	1409	.022	11.0	3780	.001194
36	.1	396.0	1.104	1274	.022	8.0	3780	.001194
37	.1	402.0	1.104	1139	.022	5.0	3780	.001194
38	.1	408.0	1.104	1004	.022	2.0	3780	.001194
39	.1	414.0	1.104	869	.022	0.0	3780	.001194
40	.1	420.0	1.104	734	.022	0.0	3780	.001194
41	.1	426.0	1.104	599	.022	0.0	3780	.001194
42	.1	432.0	1.104	464	.022	0.0	3780	.001194
43	.1	438.0	1.104	329	.022	0.0	3780	.001194
44	.1	444.0	1.104	194	.022	0.0	3780	.001194
45	.1	450.0	1.104	59	.022	0.0	3780	.001194

PT X/D STATIC PRESS. (PSIA) TW/TB TB (F) PRESS. DIFF. (PSI) FRICTION FACTOR
 1 .5 74.5 1.05 71.2 1.05 0.02
 2 90.0 1.15 71.2 1.15 68.5 0.01

AVERAGE BULK REYNOLDS 33146. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 43344. AVERAGE FRICTION FACTOR .00509

RUN 160H, DATE 04/08/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 74.5 F, TOUT = 319.2 F, MASS FLOW RATE = 20.1 LB/HR, I = 75.2 A, E = 4.40E VOLTS
 PR.IN = .72C, GR/RESS = .491E-03, MACH(2) = .157, MACH(10) = .198, T.SUR = 76.0 F

TC	X/D	T4 (F)	TW/TB	BULK REYNOLDS	HL/GAS	BULK NUSSLELT	CGAS BTU/HR FT2	Q+
1	.1	144.6	1.114	5628	.119	340.4	3420	.001083
2	.1	219.3	1.133	5588	.071	138.0	3610	.001161
3	.1	230.2	1.103	5546	.040	121.0	3721	.001177
4	.1	242.2	1.104	5467	.033	109.8	3762	.001189
5	.1	255.3	1.104	5315	.022	100.0	3775	.001193
6	.1	266.4	1.104	5175	.022	95.0	3780	.001194
7	.1	276.4	1.104	5047	.022	92.0	3780	.001194
8	.1	286.4	1.104	4913	.022	89.0	3780	.001194
9	.1	296.4	1.104	4780	.022	86.0	3780	.001194
10	.1	306.4	1.104	4648	.022	83.0	3780	.001194
11	.1	316.4	1.104	4513	.022	80.0	3780	.001194
12	.1	326.4	1.104	4379	.022	77.0	3780	.001194
13	.1	336.4	1.104	4244	.022	74.0	3780	.001194
14	.1	346.4	1.104	4109	.022	71.0	3780	.001194
15	.1	356.4	1.104	3974	.022	68.0	3780	.001194
16	.1	366.4	1.104	3839	.022	65.0	3780	.001194
17	.1	376.4	1.104	3704	.022	62.0	3780	.001194
18	.1	386.4	1.104	3569	.022	59.0	3780	.001194
19	.1	396.4	1.104	3434	.022	56.0	3780	.001194
20	.1	406.4	1.104	3299	.022	53.0	3780	.001194
21	.1	416.4	1.104	3164	.022	50.0	3780	.001194
22	.1	426.4	1.104	3029	.022	47.0	3780	.001194
23	.1	436.4	1.104	2894	.022	44.0	3780	.001194
24	.1	446.4	1.104	2759	.022	41.0	3780	.001194
25	.1	456.4	1.104	2624	.022	38.0	3780	.001194
26	.1	466.4	1.104	2489	.022	35.0	3780	.001194
27	.1	476.4	1.104	2354	.022	32.0	3780	.001194
28	.1	486.4	1.104	2219	.022	29.0	3780	.001194
29	.1	496.4	1.104	2084	.022	26.0	3780	.001194
30	.1	506.4	1.104	1949	.022	23.0	3780	.001194
31	.1	516.4	1.104	1814	.022	20.0	3780	.001194
32	.1	526.4	1.104	1679	.022	17.0	3780	.001194
33	.1	536.4	1.104	1544	.022	14.0	3780	.001194
34	.1	546.4	1.104	1409	.022	11.0	3780	.001194
35	.1	556.4	1.104	1274	.022	8.0	3780	.001194
36	.1	566.4	1.104	1139	.022	5.0	3780	.001194
37	.1	576.4	1.104	1004	.022	2.0	3780	.001194
38	.1	586.4	1.104	869	.022	0.0	3780	.001194
39	.1	596.4	1.104	734	.022	0.0	3780	.001194
40	.1	606.4	1.104	599	.022	0.0	3780	.001194
41	.1	616.4	1.104	464	.022	0.0	3780	.001194
42	.1	626.4	1.104	329	.022	0.0	3780	.001194
43	.1	636.4	1.104	194	.022	0.0	3780	.001194
44	.1	646.4	1.104	59	.022	0.0	3780	.001194

PT X/D STATIC PRESS. (PSIA) TW/TB TB (F) PRESS. DIFF. (PSI) FRICTION FACTOR
 1 .5 74.5 1.11 72.4 1.11 0.02
 2 90.1 70.2 1.29 248.5 0.15

AVERAGE BULK REYNOLDS 49783. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 37493. AVERAGE FRICTION FACTOR .00513

RUN 1514, DATE 04/08/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 75.4 F, TOUT = 443.3 F, MASS FLOW RATE = 20.0 LB/HR, I = 97.6 A/PS, E = 5.725 VOLTS
 PR. IN = .720, GR/RESQ = .753E-03, MACH(2) = .156, MACH(10) = .214, T. SUR = 30.3 F

TC	X/D	T (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	GGAS STU/HR FT2	0*
2	.1	235.8	1.213	33787	.138	338.30	71310.5	.CC2251
23	.2	383.5	1.366	55319	.208	151.00	37834.9	.CC2253
24	.1	470.7	1.704	44040	.072	135.40	76485.9	.CC2250
3	.1	549.7	1.813	34073	.453	115.00	78833.2	.CC2256
4	.1	624.9	1.884	32541	.446	101.40	79743.5	.CC2257
5	.1	710.4	1.945	40627	.347	88.80	30145.0	.CC2252
6	.1	770.3	1.945	47167	.380	81.00	30210.0	.CC2251
7	.1	815.8	1.799	45112	.063	76.40	30220.0	.CC2252
8	.1	860.0	1.751	43155	.058	72.00	30110.0	.CC2250
9	.1	899.9	1.735	41502	.061	70.00	30110.0	.CC2250
10	.1	935.5	1.681	40009	.063	73.00	30140.0	.CC2250
11	.1	967.4	1.633	38674	.064	67.00	70917.7	.CC2252
12	.1	994.7	1.585	37336	.073	67.00	70917.7	.CC2252
13	.1	1014.7	1.537	36270	.081	66.00	70917.7	.CC2252
14	.1	1030.9	1.487	35182	.126	57.00	75018.3	.CC2250
15	.1	1047.4	1.439	34242	.222	57.00	51207.3	.CC2253
16	.1	1060.7	1.390	33070	.362	57.00	43121.0	.CC2257
17	.1	1070.0	1.341	32070	.462	55.00	31211.0	.CC2257
18	.1	1077.0	1.292	31170	.557	53.00	27659.4	.CC2257
19	.1	1082.0	1.243	30320	.652	53.00	27659.4	.CC2257
20	.1	1085.0	1.194	29520	.75	53.00	27659.4	.CC2257
21	.1	1087.0	1.145	28770	.846	53.00	27659.4	.CC2257
22	.1	1088.0	1.096	28070	.941	53.00	27659.4	.CC2257
23	.1	1088.0	1.047	27420	1.035	53.00	27659.4	.CC2257
24	.1	1087.0	1.0	26820	1.128	53.00	27659.4	.CC2257
25	.1	1085.0	0.95	26270	1.22	53.00	27659.4	.CC2257
26	.1	1082.0	0.9	25770	1.31	53.00	27659.4	.CC2257
27	.1	1078.0	0.85	25320	1.39	53.00	27659.4	.CC2257
28	.1	1073.0	0.8	24920	1.47	53.00	27659.4	.CC2257
29	.1	1067.0	0.75	24570	1.55	53.00	27659.4	.CC2257
30	.1	1060.0	0.7	24270	1.63	53.00	27659.4	.CC2257
31	.1	1052.0	0.65	23970	1.71	53.00	27659.4	.CC2257
32	.1	1043.0	0.6	23720	1.79	53.00	27659.4	.CC2257
33	.1	1033.0	0.55	23520	1.87	53.00	27659.4	.CC2257
34	.1	1022.0	0.5	23370	1.95	53.00	27659.4	.CC2257
35	.1	1010.0	0.45	23270	2.03	53.00	27659.4	.CC2257
36	.1	1000.0	0.4	23220	2.11	53.00	27659.4	.CC2257
37	.1	988.0	0.35	23220	2.19	53.00	27659.4	.CC2257
38	.1	975.0	0.3	23270	2.27	53.00	27659.4	.CC2257
39	.1	962.0	0.25	23370	2.35	53.00	27659.4	.CC2257
40	.1	948.0	0.2	23520	2.43	53.00	27659.4	.CC2257
41	.1	933.0	0.15	23720	2.51	53.00	27659.4	.CC2257
42	.1	918.0	0.1	24020	2.59	53.00	27659.4	.CC2257
43	.1	902.0	0.05	24420	2.67	53.00	27659.4	.CC2257
44	.1	885.0	0.0	24920	2.75	53.00	27659.4	.CC2257
45	.1	868.0	0.0	25520	2.83	53.00	27659.4	.CC2257
46	.1	850.0	0.0	26220	2.91	53.00	27659.4	.CC2257
47	.1	832.0	0.0	27020	2.99	53.00	27659.4	.CC2257
48	.1	813.0	0.0	27920	3.07	53.00	27659.4	.CC2257
49	.1	794.0	0.0	28920	3.15	53.00	27659.4	.CC2257
50	.1	775.0	0.0	30020	3.23	53.00	27659.4	.CC2257
51	.1	756.0	0.0	31220	3.31	53.00	27659.4	.CC2257
52	.1	737.0	0.0	32520	3.39	53.00	27659.4	.CC2257
53	.1	718.0	0.0	33920	3.47	53.00	27659.4	.CC2257
54	.1	700.0	0.0	35420	3.55	53.00	27659.4	.CC2257
55	.1	682.0	0.0	37020	3.63	53.00	27659.4	.CC2257
56	.1	664.0	0.0	38720	3.71	53.00	27659.4	.CC2257
57	.1	646.0	0.0	40520	3.79	53.00	27659.4	.CC2257
58	.1	628.0	0.0	42420	3.87	53.00	27659.4	.CC2257
59	.1	610.0	0.0	44420	3.95	53.00	27659.4	.CC2257
60	.1	592.0	0.0	46520	4.03	53.00	27659.4	.CC2257
61	.1	574.0	0.0	48720	4.11	53.00	27659.4	.CC2257
62	.1	556.0	0.0	51020	4.19	53.00	27659.4	.CC2257
63	.1	538.0	0.0	53520	4.27	53.00	27659.4	.CC2257
64	.1	520.0	0.0	56120	4.35	53.00	27659.4	.CC2257
65	.1	502.0	0.0	58820	4.43	53.00	27659.4	.CC2257
66	.1	484.0	0.0	61620	4.51	53.00	27659.4	.CC2257
67	.1	466.0	0.0	64520	4.59	53.00	27659.4	.CC2257
68	.1	448.0	0.0	67520	4.67	53.00	27659.4	.CC2257
69	.1	430.0	0.0	70620	4.75	53.00	27659.4	.CC2257
70	.1	412.0	0.0	73820	4.83	53.00	27659.4	.CC2257
71	.1	394.0	0.0	77120	4.91	53.00	27659.4	.CC2257
72	.1	376.0	0.0	80520	4.99	53.00	27659.4	.CC2257
73	.1	358.0	0.0	84020	5.07	53.00	27659.4	.CC2257
74	.1	340.0	0.0	87620	5.15	53.00	27659.4	.CC2257
75	.1	322.0	0.0	91320	5.23	53.00	27659.4	.CC2257
76	.1	304.0	0.0	95120	5.31	53.00	27659.4	.CC2257
77	.1	286.0	0.0	99020	5.39	53.00	27659.4	.CC2257
78	.1	268.0	0.0	103020	5.47	53.00	27659.4	.CC2257
79	.1	250.0	0.0	107120	5.55	53.00	27659.4	.CC2257
80	.1	232.0	0.0	111320	5.63	53.00	27659.4	.CC2257
81	.1	214.0	0.0	115620	5.71	53.00	27659.4	.CC2257
82	.1	196.0	0.0	120020	5.79	53.00	27659.4	.CC2257
83	.1	178.0	0.0	124520	5.87	53.00	27659.4	.CC2257
84	.1	160.0	0.0	129120	5.95	53.00	27659.4	.CC2257
85	.1	142.0	0.0	133820	6.03	53.00	27659.4	.CC2257
86	.1	124.0	0.0	138620	6.11	53.00	27659.4	.CC2257
87	.1	106.0	0.0	143520	6.19	53.00	27659.4	.CC2257
88	.1	88.0	0.0	148520	6.27	53.00	27659.4	.CC2257
89	.1	70.0	0.0	153620	6.35	53.00	27659.4	.CC2257
90	.1	52.0	0.0	158820	6.43	53.00	27659.4	.CC2257
91	.1	34.0	0.0	164120	6.51	53.00	27659.4	.CC2257
92	.1	16.0	0.0	169520	6.59	53.00	27659.4	.CC2257
93	.1	0.0	0.0	175020	6.67	53.00	27659.4	.CC2257
94	.1	0.0	0.0	180620	6.75	53.00	27659.4	.CC2257
95	.1	0.0	0.0	186320	6.83	53.00	27659.4	.CC2257
96	.1	0.0	0.0	192120	6.91	53.00	27659.4	.CC2257
97	.1	0.0	0.0	198020	6.99	53.00	27659.4	.CC2257
98	.1	0.0	0.0	204020	7.07	53.00	27659.4	.CC2257
99	.1	0.0	0.0	210120	7.15	53.00	27659.4	.CC2257
100	.1	0.0	0.0	216320	7.23	53.00	27659.4	.CC2257

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	PERFECT
1	.5	76.7	1.17	74.3	.203
2	90.3	69.5	1.34	49.8	.203

AVERAGE BULK REYNOLDS 47355. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 32532. AVERAGE FRICTION FACTOR .00525

RUN 1624, DATE 04/08/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 76.8 F, TOUT = 336.5 F, MASS FLOW RATE = 20.0 LB/HR, I = 114.8 A/PS, E = 6.390 VOLTS
 PR. IN = .719, GR/RESQ = .103E-02, MACH(2) = .157, MACH(10) = .234, T. SUR = 34.7 F

TC	X/D	T (F)	TW/TB	BULK REYNOLDS	HL/QGAS	BULK NUSSELT	GGAS STU/HR FT2	0*
2	.1	235.8	1.298	33787	.138	338.30	71310.5	.CC2251
23	.2	383.5	1.366	55319	.208	151.00	37834.9	.CC2253
24	.1	470.7	1.704	44040	.072	135.40	76485.9	.CC2250
3	.1	549.7	1.813	34073	.453	115.00	78833.2	.CC2256
4	.1	624.9	1.884	32541	.446	101.40	79743.5	.CC2257
5	.1	710.4	1.945	40627	.347	88.80	30145.0	.CC2252
6	.1	770.3	1.945	47167	.380	81.00	30210.0	.CC2251
7	.1	815.8	1.799	45112	.063	76.40	30220.0	.CC2252
8	.1	860.0	1.751	43155	.058	72.00	30110.0	.CC2250
9	.1	899.9	1.735	41502	.061	70.00	30110.0	.CC2250
10	.1	935.5	1.681	40009	.063	73.00	30140.0	.CC2250
11	.1	967.4	1.633	38674	.064	67.00	70917.7	.CC2252
12	.1	994.7	1.585	37336	.073	67.00	70917.7	.CC2252
13	.1	1014.7	1.537	36270	.081	66.00	70917.7	.CC2252
14	.1	1030.9	1.487	35182	.126	57.00	75018.3	.CC2250
15	.1	1047.4	1.439	34242	.222	57.00	51207.3	.CC2257
16	.1	1060.7	1.390	33070	.362	57.00	43121.0	.CC2257
17	.1	1070.0	1.341	32070	.462	55.00	31211.0	.CC2257
18	.1	1077.0	1.292	31170	.557	53.00	27659.4	.CC2257
19	.1	1082.0	1.243	30320	.652	53.00	27659.4	.CC2257
20	.1	1085.0	1.194	29520	.75	53.00	27659.4	.CC2257
21	.1	1087.0	1.145	28770	.846	53.00	27659.4	.CC2257
22	.1	1088.0	1.096	28070	.941	53.00	27659.4	.CC2257
23	.1	1087.0	1.047	27420	1.035	53.00	27659.4	.CC2257
24	.1	1085.0	1.0	26820	1.128	53.00	27659.4	.CC2257
25	.1	1082.0	0.95	26270	1.22	53.00	27659.4	.CC2257
26	.1	1078.0	0.9	25770	1.31	53.00	27659.4	.CC2257
27	.1	1073.0	0.85	25320	1.39	53.00	27659.4	.CC2257
28	.1	1067.0	0.8	24920	1.47	53.00	276	

RUN 1634, DATE 04/08/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 76.8 F, TCUT = 345.6 F, MASS FLOW RATE = 11.9 LB/HR, I = 64.4 AMP, E = 3.635 VOLTS
 PR. IN = .719, GR/RESS = .572E-03, MACH(2) = .116, MACH(16) = .145, T. SURR = 93.0 F

TC	X/D	T _w (F)	T _w /T _b	BULK REYNOLDS	HL/2GAS	BULK NUSSELT	2GAS BTU/HR FT ²	3+
1.0	.1	152.0	1.190	32277.0	.201	104.33	169.71	.001133
1.2	.2	222.0	1.136	32277.0	.252	104.33	169.71	.001133
1.4	.4	322.0	1.067	32277.0	.364	104.33	169.71	.001133
1.6	.6	444.0	1.000	32277.0	.500	104.33	169.71	.001133
1.8	.8	577.0	0.944	32277.0	.667	104.33	169.71	.001133
2.0	1.0	722.0	0.899	32277.0	.867	104.33	169.71	.001133
2.2	1.2	877.0	0.863	32277.0	1.100	104.33	169.71	.001133
2.4	1.4	1042.0	0.833	32277.0	1.367	104.33	169.71	.001133
2.6	1.6	1217.0	0.807	32277.0	1.667	104.33	169.71	.001133
2.8	1.8	1402.0	0.784	32277.0	2.000	104.33	169.71	.001133
3.0	2.0	1597.0	0.763	32277.0	2.367	104.33	169.71	.001133
3.2	2.2	1802.0	0.743	32277.0	2.767	104.33	169.71	.001133
3.4	2.4	2017.0	0.724	32277.0	3.200	104.33	169.71	.001133
3.6	2.6	2242.0	0.706	32277.0	3.667	104.33	169.71	.001133
3.8	2.8	2477.0	0.689	32277.0	4.167	104.33	169.71	.001133
4.0	3.0	2722.0	0.673	32277.0	4.700	104.33	169.71	.001133
4.2	3.2	2977.0	0.658	32277.0	5.267	104.33	169.71	.001133
4.4	3.4	3242.0	0.644	32277.0	5.867	104.33	169.71	.001133
4.6	3.6	3517.0	0.630	32277.0	6.500	104.33	169.71	.001133
4.8	3.8	3802.0	0.617	32277.0	7.167	104.33	169.71	.001133
5.0	4.0	4097.0	0.604	32277.0	7.867	104.33	169.71	.001133
5.2	4.2	4402.0	0.592	32277.0	8.600	104.33	169.71	.001133
5.4	4.4	4717.0	0.580	32277.0	9.367	104.33	169.71	.001133
5.6	4.6	5042.0	0.569	32277.0	10.167	104.33	169.71	.001133
5.8	4.8	5377.0	0.558	32277.0	11.000	104.33	169.71	.001133
6.0	5.0	5722.0	0.548	32277.0	11.867	104.33	169.71	.001133
6.2	5.2	6077.0	0.538	32277.0	12.767	104.33	169.71	.001133
6.4	5.4	6442.0	0.529	32277.0	13.700	104.33	169.71	.001133
6.6	5.6	6817.0	0.520	32277.0	14.667	104.33	169.71	.001133
6.8	5.8	7202.0	0.512	32277.0	15.667	104.33	169.71	.001133
7.0	6.0	7597.0	0.504	32277.0	16.700	104.33	169.71	.001133
7.2	6.2	8002.0	0.496	32277.0	17.767	104.33	169.71	.001133
7.4	6.4	8417.0	0.489	32277.0	18.867	104.33	169.71	.001133
7.6	6.6	8842.0	0.482	32277.0	20.000	104.33	169.71	.001133
7.8	6.8	9277.0	0.475	32277.0	21.167	104.33	169.71	.001133
8.0	7.0	9722.0	0.468	32277.0	22.367	104.33	169.71	.001133
8.2	7.2	10177.0	0.462	32277.0	23.600	104.33	169.71	.001133
8.4	7.4	10642.0	0.456	32277.0	24.867	104.33	169.71	.001133
8.6	7.6	11117.0	0.450	32277.0	26.167	104.33	169.71	.001133
8.8	7.8	11602.0	0.444	32277.0	27.500	104.33	169.71	.001133
9.0	8.0	12097.0	0.438	32277.0	28.867	104.33	169.71	.001133
9.2	8.2	12602.0	0.433	32277.0	30.267	104.33	169.71	.001133
9.4	8.4	13117.0	0.428	32277.0	31.700	104.33	169.71	.001133
9.6	8.6	13642.0	0.423	32277.0	33.167	104.33	169.71	.001133
9.8	8.8	14177.0	0.418	32277.0	34.667	104.33	169.71	.001133
10.0	9.0	14722.0	0.414	32277.0	36.200	104.33	169.71	.001133

PT	X/D	STATIC PRESS. (PSIA)	T _w /T _b	T _b (F)	PRESS CORRECT
1	.5	59.2	1.12	79.3	.02
2	90.1	57.1	1.28	329.5	.01

AVERAGE BULK REYNOLDS 28730. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 21742. AVERAGE FRICTION FACTOR .00612

RUN 1644, DATE 04/08/76, GAS AIR, MOLECULAR WT. = 29.07
 TIN = 77.2 F, TCUT = 473.1 F, MASS FLOW RATE = 11.9 LB/HR, I = 79.0 AMP, E = 4.565 VOLTS
 PR. IN = .719, GR/RESS = .644E-03, MACH(2) = .117, MACH(16) = .159, T. SURR = 84.0 F

TC	X/D	T _w (F)	T _w /T _b	BULK REYNOLDS	HL/2GAS	BULK NUSSELT	2GAS BTU/HR FT ²	3+
1.0	.1	170.0	1.190	32277.0	.175	219.63	325.30	.001734
1.2	.2	280.0	1.136	32277.0	.291	102.21	325.30	.001734
1.4	.4	333.0	1.067	32277.0	.406	93.69	325.30	.001734
1.6	.6	395.0	1.000	32277.0	.521	80.22	325.30	.001734
1.8	.8	443.0	0.944	32277.0	.637	70.86	325.30	.001734
2.0	1.0	509.0	0.899	32277.0	.754	62.80	325.30	.001734
2.2	1.2	550.0	0.863	32277.0	.870	53.65	325.30	.001734
2.4	1.4	613.0	0.833	32277.0	1.022	45.22	325.30	.001734
2.6	1.6	679.0	0.807	32277.0	1.167	36.50	325.30	.001734
2.8	1.8	749.0	0.784	32277.0	1.344	27.50	325.30	.001734
3.0	2.0	823.0	0.763	32277.0	1.544	18.50	325.30	.001734
3.2	2.2	901.0	0.743	32277.0	1.767	9.50	325.30	.001734
3.4	2.4	983.0	0.724	32277.0	2.000	0.50	325.30	.001734
3.6	2.6	1069.0	0.706	32277.0	2.250	0.00	325.30	.001734
3.8	2.8	1159.0	0.689	32277.0	2.517	0.00	325.30	.001734
4.0	3.0	1253.0	0.673	32277.0	2.800	0.00	325.30	.001734
4.2	3.2	1351.0	0.658	32277.0	3.100	0.00	325.30	.001734
4.4	3.4	1453.0	0.644	32277.0	3.417	0.00	325.30	.001734
4.6	3.6	1559.0	0.630	32277.0	3.750	0.00	325.30	.001734
4.8	3.8	1669.0	0.617	32277.0	4.100	0.00	325.30	.001734
5.0	4.0	1783.0	0.604	32277.0	4.467	0.00	325.30	.001734
5.2	4.2	1901.0	0.592	32277.0	4.850	0.00	325.30	.001734
5.4	4.4	2023.0	0.580	32277.0	5.250	0.00	325.30	.001734
5.6	4.6	2149.0	0.569	32277.0	5.667	0.00	325.30	.001734
5.8	4.8	2279.0	0.558	32277.0	6.100	0.00	325.30	.001734
6.0	5.0	2413.0	0.548	32277.0	6.550	0.00	325.30	.001734
6.2	5.2	2551.0	0.538	32277.0	7.017	0.00	325.30	.001734
6.4	5.4	2693.0	0.529	32277.0	7.500	0.00	325.30	.001734
6.6	5.6	2839.0	0.520	32277.0	8.000	0.00	325.30	.001734
6.8	5.8	2989.0	0.512	32277.0	8.517	0.00	325.30	.001734
7.0	6.0	3143.0	0.504	32277.0	9.050	0.00	325.30	.001734
7.2	6.2	3301.0	0.496	32277.0	9.600	0.00	325.30	.001734
7.4	6.4	3463.0	0.489	32277.0	10.167	0.00	325.30	.001734
7.6	6.6	3629.0	0.482	32277.0	10.750	0.00	325.30	.001734
7.8	6.8	3799.0	0.475	32277.0	11.350	0.00	325.30	.001734
8.0	7.0	3973.0	0.468	32277.0	11.967	0.00	325.30	.001734
8.2	7.2	4151.0	0.462	32277.0	12.600	0.00	325.30	.001734
8.4	7.4	4333.0	0.456	32277.0	13.250	0.00	325.30	.001734
8.6	7.6	4519.0	0.450	32277.0	13.917	0.00	325.30	.001734
8.8	7.8	4709.0	0.444	32277.0	14.600	0.00	325.30	.001734
9.0	8.0	4903.0	0.438	32277.0	15.300	0.00	325.30	.001734
9.2	8.2	5101.0	0.433	32277.0	16.017	0.00	325.30	.001734
9.4	8.4	5303.0	0.428	32277.0	16.750	0.00	325.30	.001734
9.6	8.6	5509.0	0.423	32277.0	17.500	0.00	325.30	.001734
9.8	8.8	5719.0	0.418	32277.0	18.267	0.00	325.30	.001734
10.0	9.0	5933.0	0.414	32277.0	19.050	0.00	325.30	.001734

PT	X/D	STATIC PRESS. (PSIA)	T _w /T _b	T _b (F)	PRESS CORRECT
1	.5	59.1	1.15	77.2	.02
2	90.3	56.6	1.33	451.3	.01

AVERAGE BULK REYNOLDS 27706. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS AVERAGE WALL REYNOLDS 19706. AVERAGE FRICTION FACTOR .00390

RUN 1654, DATE 04/04/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 77.7 F, TOUT = 523.1 F, MASS FLOW RATE = 11.8 LB/HR, I = 93.0 A, E = 5.533 VOLTS
 PR. IN = .719, GR/RES = .116E-02, MACH(2) = .117, MACH(16) = .171, T. SUR = 558 F

TC	X/D	T (F)	TW/TB	BULK REYNOLDS	HL/OGAS	BULK NUSSELT	GGAS BTU/HR FT ²	G+
2	.1	220.1	1.262	3328	.174	100.0	1.1	00220
3	.2	361.1	1.510	3330	.300	100.0	1.1	00361
4	.4	443.3	1.647	3333	.410	100.0	1.1	00443
5	.6	533.3	1.752	3333	.500	100.0	1.1	00533
6	.8	613.3	1.838	3333	.570	100.0	1.1	00613
7	1.0	704.4	1.941	3333	.630	100.0	1.1	00704
8	1.2	763.3	2.008	3333	.680	100.0	1.1	00763
9	1.4	809.9	2.041	3333	.720	100.0	1.1	00809
10	1.6	840.0	2.091	3333	.770	100.0	1.1	00840
11	1.8	852.2	2.142	3333	.810	100.0	1.1	00852
12	2.0	857.7	2.193	3333	.850	100.0	1.1	00857
13	2.2	859.9	2.244	3333	.890	100.0	1.1	00859
14	2.4	859.9	2.295	3333	.930	100.0	1.1	00859
15	2.6	859.9	2.346	3333	.970	100.0	1.1	00859
16	2.8	859.9	2.397	3333	1.010	100.0	1.1	00859
17	3.0	859.9	2.448	3333	1.050	100.0	1.1	00859

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	POSITIVE CORRECT
1	.5	59.1	1.22	78.2	-02
2	0.4	56.0	1.40	55.5	+01

AVERAGE BULK REYNOLDS 26540. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 17482. AVERAGE FRICTION FACTOR .0013

RUN 1674, DATE 04/03/76, GAS AIR, MOLECULAR WT. = 28.97
 TIN = 75.4 F, TOUT = 177.1 F, MASS FLOW RATE = 11.3 LB/HR, I = 39.3 A, E = 2.310 VOLTS
 PR. IN = .720, GR/RES = .216E-03, MACH(2) = .117, MACH(16) = .127, T. SUR = 50.0 F

TC	X/D	T (F)	TW/TB	BULK REYNOLDS	HL/OGAS	BULK NUSSELT	GGAS BTU/HR FT ²	G+
2	.1	99.9	1.047	3312	.109	247.28	1.1	00099
3	.2	129.0	1.094	3312	.273	105.44	1.1	00129
4	.4	140.0	1.120	3300	.491	96.19	1.1	00140
5	.6	154.6	1.141	3292	.651	93.70	1.1	00154
6	.8	169.5	1.154	3272	.807	76.53	1.1	00169
7	1.0	180.0	1.161	3233	.935	71.44	1.1	00180
8	1.2	190.0	1.161	3193	1.036	69.66	1.1	00190
9	1.4	199.3	1.159	3153	1.137	69.30	1.1	00199
10	1.6	207.7	1.156	3122	1.239	67.40	1.1	00207
11	1.8	216.7	1.154	3092	1.341	66.33	1.1	00216
12	2.0	222.9	1.154	3062	1.444	66.72	1.1	00222
13	2.2	229.8	1.149	3032	1.545	66.71	1.1	00229
14	2.4	233.3	1.148	3002	1.649	66.35	1.1	00233
15	2.6	235.3	1.145	2972	1.756	66.00	1.1	00235
16	2.8	237.3	1.143	2942	1.863	65.77	1.1	00237
17	3.0	239.3	1.142	2912	1.974	65.76	1.1	00239

PT	X/D	STATIC PRESS. (PSIA)	TW/TB	TB (F)	POSITIVE CORRECT
1	.5	59.0	1.04	74.4	-02
2	0.6	57.5	1.13	64.4	+01

AVERAGE BULK REYNOLDS 31175. AVERAGE PARAMETERS BETWEEN PRESSURE TAPS
 AVERAGE WALL REYNOLDS 27258. AVERAGE FRICTION FACTOR .00541

APPENDIX E.
 Thermocouple Conduction Error
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A thermocouple attached to the outside surface of a heated tube acts as a fin or extended surface which lowers the temperature at the point of attachment. Since this point also serves as the thermocouple junction, it measures a wall temperature lower than the value which would occur without its presence. The difference is known as "thermocouple conduction error." Consequently, in an experiment such as the present study, the deduced Nusselt number is systematically increased unless corrected for this effect.

Analyses

Based on extended surface analyses, the normalized thermocouple conduction error θ has been shown by Schneider [48] to be approximately

$$\theta = \frac{t_w - t_{TC}}{t_w - t_\infty} = \left[1 + \frac{2\pi k_{tube} \delta_{tube} \lambda r_o}{h_{TC} A_s} \frac{K_1(\lambda r_o)}{K_0(\lambda r_o)} \right]^{-1} \quad (E1)$$

where t_w is the temperature of the undisturbed tube, $h_{TC} A_s$ is the thermocouple conductance and r_o is the effective radius of the thermocouple attachment. K_0 and K_1 are modified Bessel functions of the second kind of order zero and one, respectively [49,50]. The quantity λ is defined as

$$\lambda = \frac{h_o + h_i}{k_{tube} \delta_{tube}}$$

and the thermocouple conductance is defined by the equation

$q_{TC} = h_{TC} A_s (t_{TC} - t_\infty)$. Thus, in this approximation θ is a function of the non-dimensional parameters λr_o and $M = k_{tube} \delta_{tube} / (h_{TC} A_s)$.

Using approximations to the Bessel functions, valid at small values of the argument, one may reduce equation (E1) further so that it takes the form

$$\theta \approx - \ln(\lambda r_o) / 2\pi M \quad (E2)$$

when $2\pi M \gg 1$. This form is useful for estimates of the magnitude of θ when desiring to determine whether it is significant in a given case. It is presented as Figure E1.

In calibration for thermocouple conduction error data are normally obtained without flow so a probe can be used to measure the tube wall temperature in the vicinity of the thermocouple. In this case, $h_i = 0$. Examining Figure E1, one can see that the effect of flow (i.e., non zero value of h_i) is to increase λr_o and reduce θ for the same thermocouple attachment and environment.

Hess [29] extended and improved Schneider's analysis for application to electrically heated tubes with internal flow such as the present experiment. His representation takes the form

$$\theta = \frac{h_i + h_o}{h_{TC}} \frac{(1 - h_o / h_{TC})}{1 + \frac{2k_w \delta}{h_{TC} r_o^2} \frac{\lambda r_o K_1(\lambda r_o)}{K_o(\lambda r_o)}} \quad (E3)$$

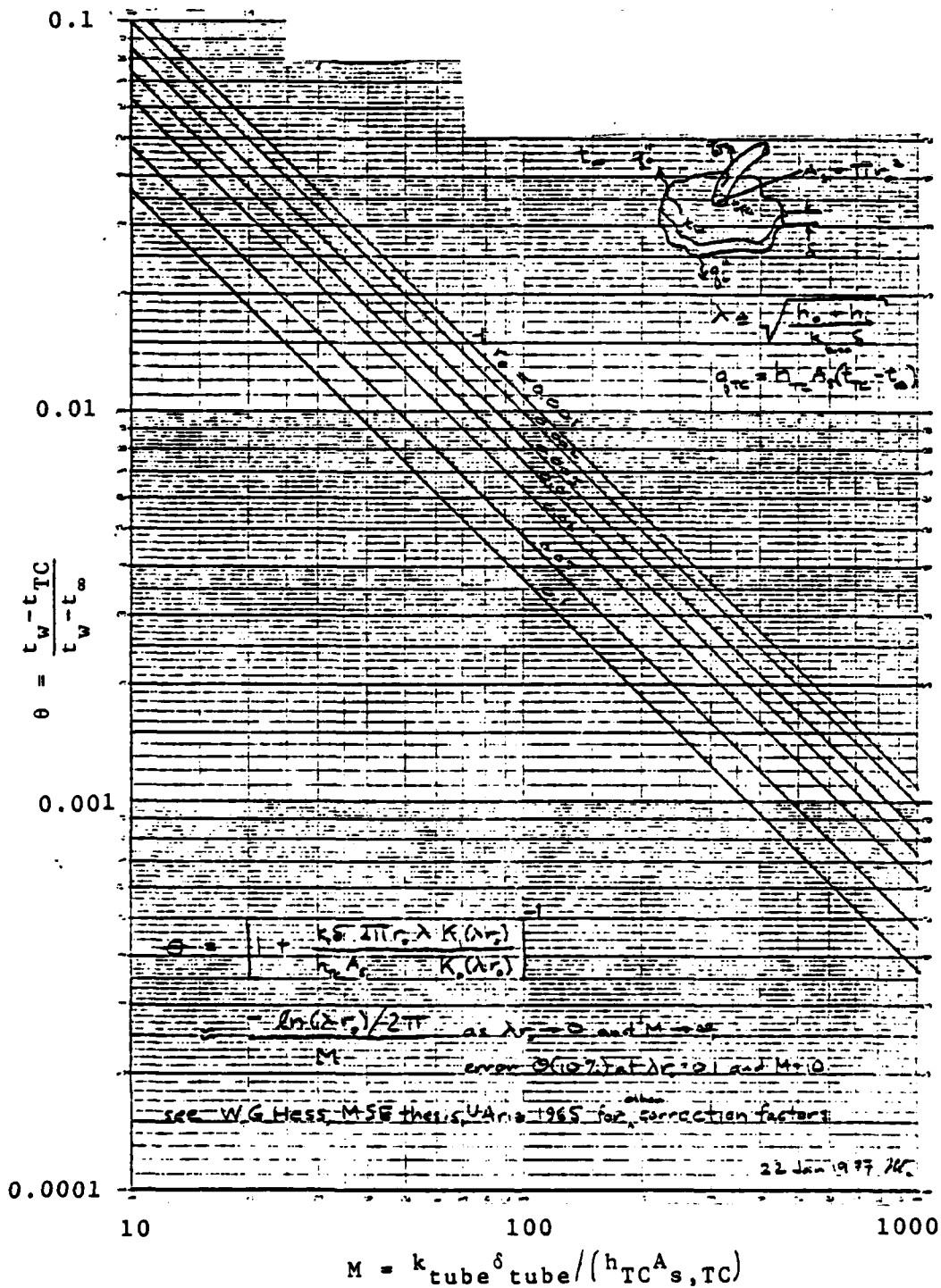


Fig. E1. Approximation to thermocouple conduction error for large values of parameter $M = k_{\text{tube}} \delta_{\text{tube}} / (h_{TC} A_{s,TC})$.

Evaluation of thermocouple conductance, $h_{TC}A_s$

For fine wire thermocouples in an environment at atmospheric pressure, free convection dominates as the mechanism for heat loss from the thermocouple compared to radiation. As an approximation, one may consider the total heat transfer coefficient and properties to be constant for the thermocouple wire and derive

$$q_{TC} = \sqrt{(h_r + h_{NC})Pk_{TC}A_s} (t_{TC} - t_\infty) \tanh ml \quad (E4)$$

as by Schneider [48]. For long wires, i.e., $ml = \sqrt{2 hP/kA} l > 5$, $\tanh ml$ approaches unity; then $h_{TC}A_s$ becomes $\sqrt{(h_r + h_{NC})Pk_{TC}A_s}$.

The wire can be considered a small body in large surroundings so

$$h_r \approx \epsilon \sigma [T_{TC}^4 - T_\infty^4] / (T_{TC} - T_\infty) \quad (E5)$$

The heat transfer coefficient for natural convection can be determined from a correlation of the form $Nu_f = f_n(Gr_f Pr)$. The Grashof number is typically small for wires of the size of our thermocouples. For the range $10^{-3} < Gr Pr < 10^{-1}$ the curve recommended by Kreith [51, Fig.7-3] can be represented as

$$Nu_f = \frac{h_{NC}d}{K_f} = 0.315 + 0.8 [Gr_f Pr_f]^{0.18} \quad (E6)$$

In an unpublished note, Hess, Deardorff and McEligot [52, included herein as Appendix F] examined available calibration data for radiating thermocouples attached in the parallel junction form of Moen [28]. From comparisons be-

tween predictions and measurements, they concluded that the effective radius of the thermocouple attachment, r_o , was approximately equal to the actual radius of the thermocouple wire. Consequently, $A_s = \pi r_o^2 \approx A_{CS} = \pi d^2/4$. The calibration data of Campbell [53] for an atmospheric environment are also in approximate agreement with the choice of $d/2$ as r_o .

Application to present experiment

The heat transfer coefficient from the outside of the tube, h_o , may be deduced from the heat loss calibration equation (B5), to be

$$h_o = [C_1 + C_2(t_w - t_\infty) + C_3(t_w - t_\infty)^2]/\pi D \quad (E7)$$

For h_i either a correlation such as equation (19) or tabular data from the initial data reduction can be employed. In this experiment h_i is typically of the order of 200 Btu/hrft²°F or more.

As an example of the magnitude of thermocouple conduction error to be expected, Figure E2 has been plotted for thermocouple 10. The value of h_i was taken as 200 Btu/hrft²°F and the environmental temperature was assumed to be about 70°F for this presentation. The reduction in θ with flow is clear and it is also seen that in the heat loss runs (no flow) θ decreases slightly as the wall temperature increases.

For the tabular results of Appendix D the thermocouple conduction error was calculated from equations (E3) through (E7) and correlations (19) and (B5). Material and fluid

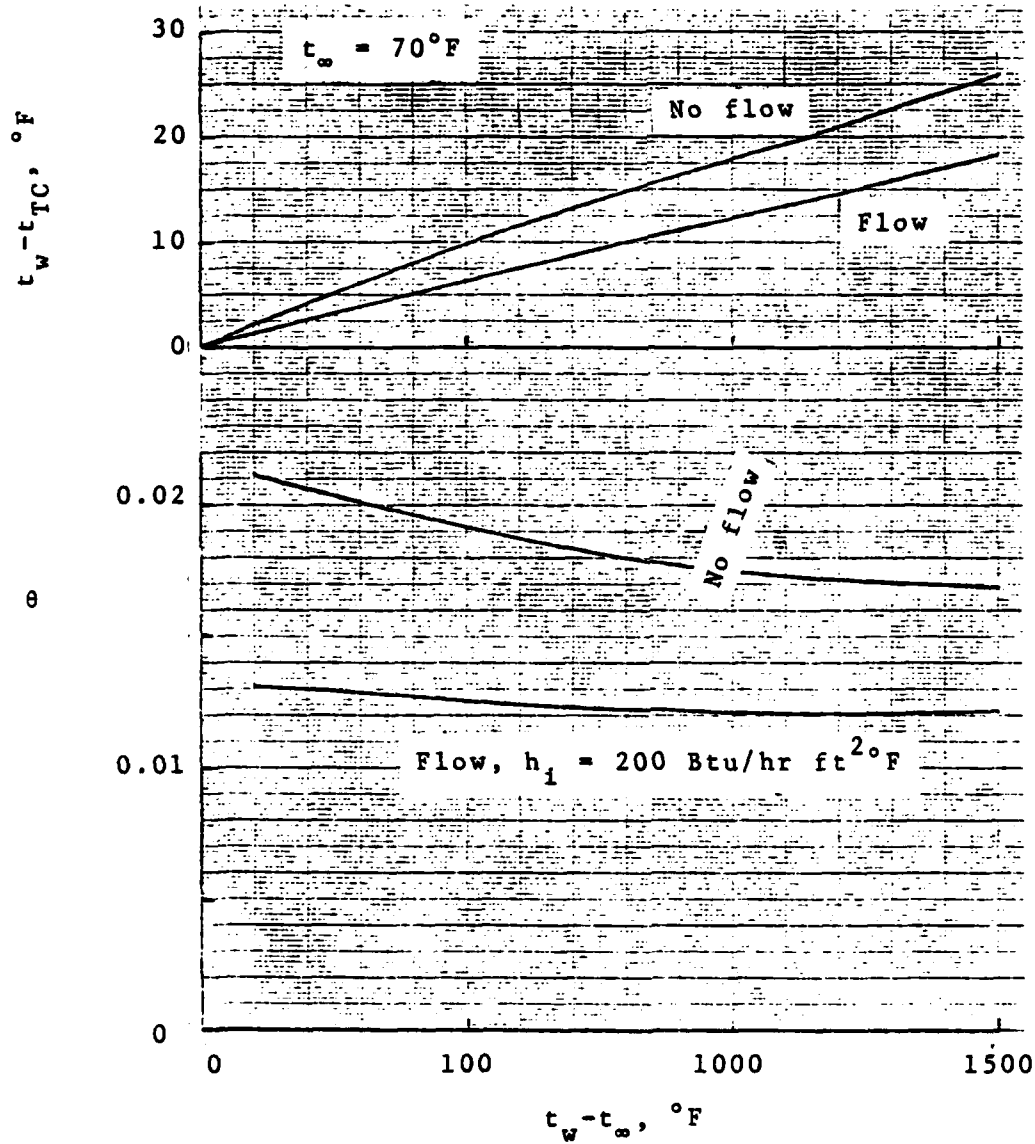


Fig. E2. Predicted thermocouple conduction error for thermocouple 10.

properties used in evaluating the thermocouple conductance were based on the temperature at the junction and its resulting film temperature, $t_f = (t_{TC} + t_\infty)/2$, as appropriate. While a numerical solution could be applied to improve the analysis predicting the thermocouple conductance, such sophistication does not appear warranted due to the uncertain knowledge of several quantities and the small magnitude of θ .

APPENDIX F.

Radiating Thermocouple Conduction Error

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In space environments and space simulation chambers, temperatures are often measured with thermocouples attached to exposed surfaces. The primary means of energy exchange then are conduction through the solid material and thermal radiation. In our Laboratory, we also often use a vacuum environment to minimize and/or localize the heat loss from thin-walled tubes in which we perform internal convective heat transfer measurements [38]. In these situations the thermocouple attachment usually acts as a radiating fin which reduces the local surface temperature near the point of measurement. This systematic effect may be called the radiating thermocouple conduction error.

Schneider [48] presents an analysis to predict the thermocouple conduction error in a convective environment by idealizing the thermocouple as a cylinder mounted perpendicular to the surface at a single point.

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2. Now with Gulf General Atomics, La Jolla, California.

Including energy generation in the wall by electrical resistive heating and energy transfer from the surface opposite the idealized thermocouple, one may extend Schneider's result to

$$\frac{T_{TC} - T_{w,u}}{T_{w,u} - T_{\infty}} = \frac{(h_o - h_{TC})r K_o(\lambda r)}{(h_i + h_{TC})r K_o(\lambda r) + 2\lambda k_w \delta K_1(\lambda r)} \quad (F1)$$

where the heat transfer coefficients may represent convective or radiative processes as appropriate. In the case of infinite radiating thermocouple leads, the effective heat transfer coefficient over the contact area of the thermocouple may be shown to be

$$h_{TC} = \frac{k_{TC} \left[\frac{2R}{5} (T_{TC}^5 - 5T_{TC}T_{\infty}^4 + 4T_{\infty}^5) \right]^{1/2}}{T_{TC} - T_{\infty}} \quad (f2)$$

if its emissivity and thermal conductivity are constant. Thus, provided that the material properties are known, prediction of the radiating thermocouple conduction error reduces to the problem of determining the effective thermocouple radius, r .

For many applications the parallel type thermocouple junction, shown in the insert of Figure F1, is more accurate than the more common cross type junction because the location of the measuring plane is effectively on the tube surface rather than being spread perpendicular to it [28].

Rather than satisfying the idealization of a single cylindrical interface between the thermocouple and the surface, the attachment region for the parallel junction consists of two roughly elliptical areas slightly separated from each other. Accordingly, the objectives of the present work were taken to be (1) to determine r for a parallel junction configuration and (2) to investigate the reproducibility of the conduction error when such thermocouples are produced by using normal laboratory standards for equipment construction.

Measurements were conducted on three circular test sections of 0.010 inch thick Inconel 600, two feet long. Premium grade bare Chromel and Alumel thermocouple wires of 0.005 inch diameter were spot welded to the test section by the electrical discharge technique. Circumferential distance between the two wires was approximately 1/8 inch and the attached area of each covered approximately one to two wire diameters. Tests included about fourteen such thermocouples with all wires taken from the same spools.

These resistively heated test sections were mounted in glass vacuum chambers. With no internal flow, h_i equals zero and h_o can be determined from the tube emissivity which one also deduces from the tests. "Undisturbed"

tube wall temperatures, $T_{w,u}$, were determined with a traveling internal thermocouple probe, also of premium grade Chromel-Alumel, which measured the wall temperature profile axially between the thermocouples. Calculations show the maximum temperature drop through the wall to be less than $0.01^\circ F$ so the thin wall idealization is valid. Readings were accepted without correction for deviation from standard N.B.S. emf tables since Hoskins Manufacturing Company certified the deviation as less than $1^\circ F$.

Results are demonstrated on Figure F1. The dashed curves are predictions based on equations (F1) and (F2) in conjunction with manufacturers' information for emissivities and thermal conductivities of the thermocouple wires and the tube. The solid curve represents predictions based on the measured emissivity of the Inconel tube used by Hess and on an effective thermocouple radius equal to the actual wire diameter; otherwise the bases are the same.

Hess' data points are averages of the thermocouple readings for the central portion of the tube and they show the effective radius to be approximately equal to the wire diameter or slightly less. The measurements of Swearingen and of Reynolds and Deardorff are from test sections with different thermal histories, hence

emissivities, but of the same materials and dimensions. Their calibrations suggest that r is about one-half the wire diameter. Different welding jigs were used for each and, consequently, the region of attachment varied from test section to test section but would be approximately uniform for different thermocouples on the same test section. Accordingly, one would expect the level of the thermocouple conduction error to vary from test section to test section as it does in Figure F1.

We conclude that the effective radius of the thermocouple attachment is approximately one-half to one wire diameter when constructed in the manner described. One may use this observation with manufacturers' information and equations (F1) and (F2) to determine whether the systematic error will be significant in his specific application. (For the results shown, in an internal convective heating experiment with $T_w \approx 1000^\circ\text{F}$ and $T_b \approx 900^\circ\text{F}$, the resulting error in Nusselt number would be 5 to 10 per cent.) If such predictions indicate that the errors would be important, we recommend individual calibration since the values of a number of the pertinent input variables are not readily available.

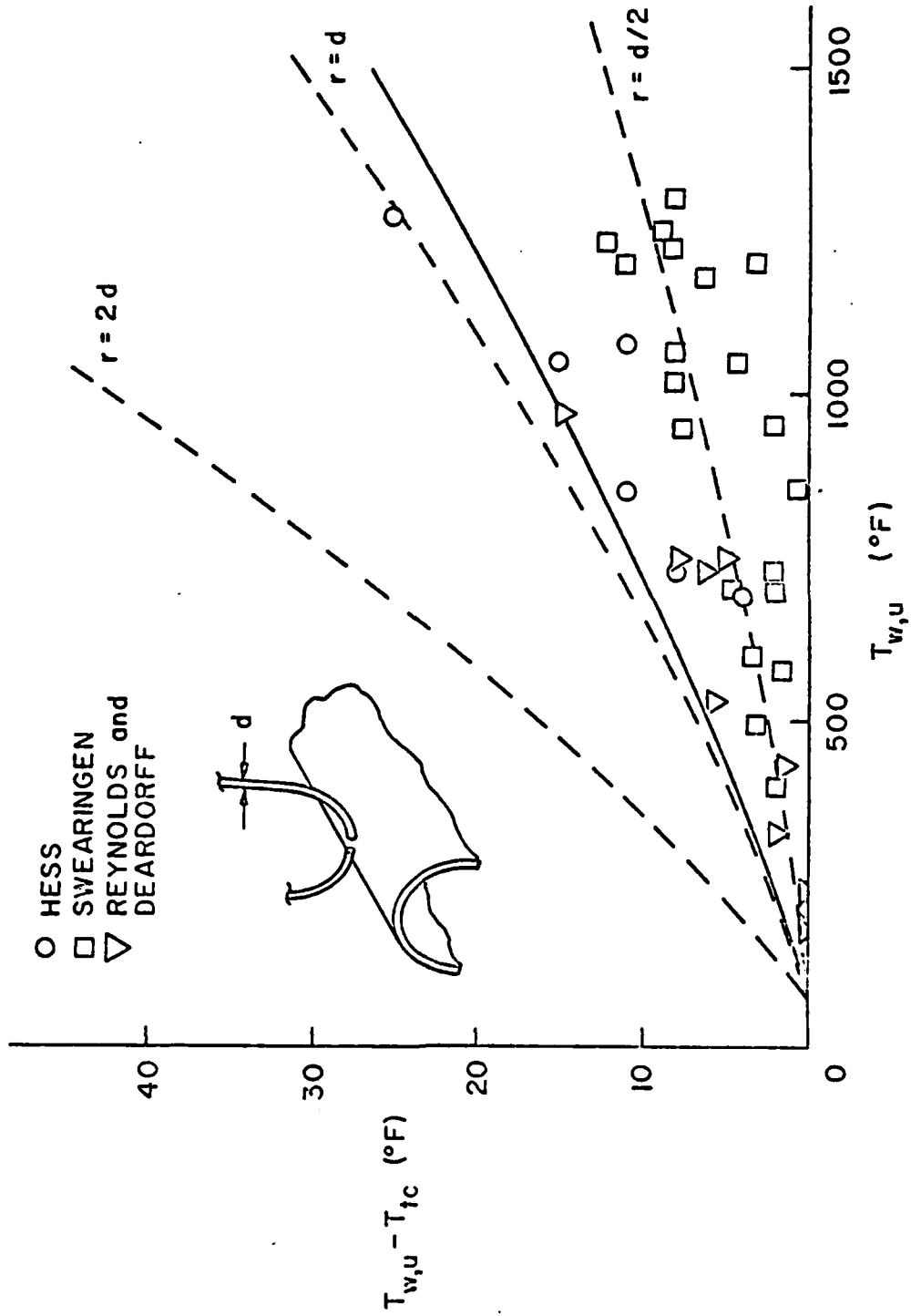


Figure F1. RADIATING THERMOCOUPLE CONDUCTION ERROR FOR PARALLEL TYPE JUNCTION

NOMENCALTURE

h	Heat transfer coefficient; h_i , inner surface: (thermocouple) surface.
k	thermal conductivity
K_0, K_1	Bessel functions
r	effective thermocouple attachment radius
T_{TC}	temperature measured by thermocouple
$T_{w,u}$	"undisturbed" wall temperature
β	thermocouple heat transfer constant, $2\sigma\epsilon/(k_{TC}r)$
δ	wall thickness
ϵ	emissivity
λ	wall heat transfer constant, $[h_o + h_i]/(k_w r)^{1/2}$
σ	Stefan-Boltzman constant

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