

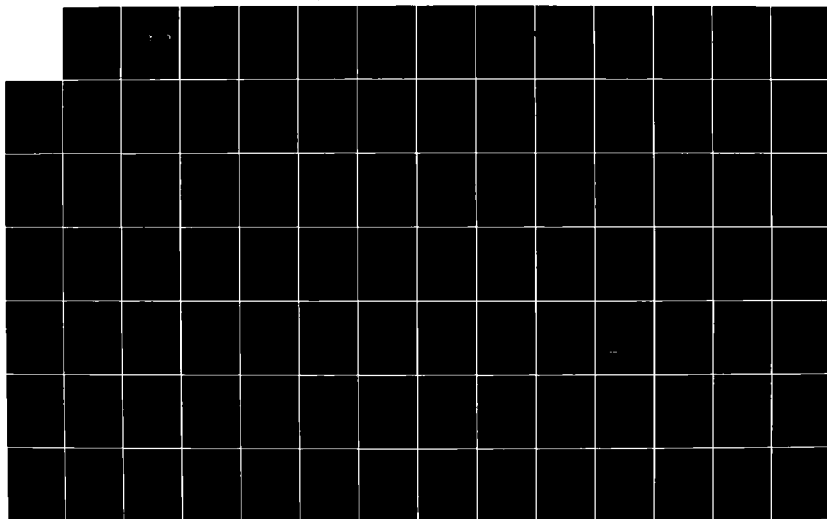
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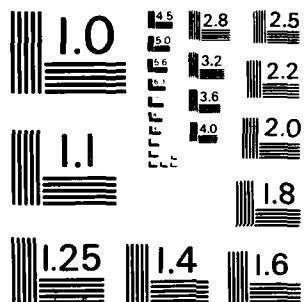
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THESIS

DUAL-PHASE NOZZLE FLOW

by

Thomas C. Nollie, Jr.

October 1982

Thesis Advisor:

J. F. Sladky

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Dual-Phase Nozzle Flow

by

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Lieutenant, United States Navy
B.S., United States Naval Academy, 1975

Submitted in partial fulfillment of the
requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

from the

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ABSTRACT

A review of the dual-phase power system was made. This study focused on the multi-component nozzle of this dual-phase system. First, an existing computer code predicting the nozzle performance was updated, and second a series of experimental tests on a variable area, two-dimensional nozzle was performed to verify the computer code.



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I. DUAL-PHASE CYCLE

The dual-phase nozzle is a key element in the dual-phase engine concept. This nozzle will be studied in detail, but first a review of the dual-phase cycle will be carried out from information obtained from References 1 and 2.

Reference 1 describes the dual-phase engine. It is a new concept which operates on a mixture of two fluids or two phases of one fluid. This cycle employs a high-torque/low rpm impulse turbine which eliminates the requirement for speed reduction needed in a conventional steam turbine. This allows for direct drive of a ship's propeller with no gear reduction. This is ideally suited for marine propulsion, reducing weight, noise, and volume.

The dual-phase concept is a modified Rankine cycle with subtle differences of extreme importance. In a normal steam turbine the working fluid enters a turbine through a nozzle where the kinetic energy of the working fluid is converted to a mechanical form. The dual-phase system introduces a second fluid prior to entry into the nozzle. This fluid is of higher vapor pressure than the steam and therefore remains in the liquid state throughout the cycle. Section A will describe this two-component cycle while section B will do the same for a single-component system. The dual-phase

system can be divided into the two groups illustrated in Figure 1. Single-component flow can be further divided into three categories.

A. TWO-COMPONENT

A two-component mixture is one in which the low vapor pressure liquid and a high vapor pressure liquid are of different chemical compounds. Some fluid combinations which have been considered are steam-krytox, steam-caloria, steam-lead, bismuth eutectic, and dow-therminol. A schematic flow diagram and process representation on the T-S diagram are shown in Figures 2 and 3 for the two-phase engine cycle using a "two-component" mixture. The liquid phase is lithium carbonate and vapor phase is steam. To illustrate the overall advantages of the two-phase engine cycle a discussion on the theory of operation will be presented using a two-component mixture and Figures 2 and 3.

The major component of the dual-phase system is the nozzle. A mixer area is located prior to the nozzle inlet. A high vapor pressure liquid is placed in contact with the low vapor pressure liquid in this area. A high pressure vapor liquid mixture is formed. Since the temperature of the liquid is greater than temperature of the water, heat is transferred to the water causing it to vaporize to point 1. Figure 4 illustrates the temperature and state point from the inlet of the mixture area to the nozzle exit. This mixture is

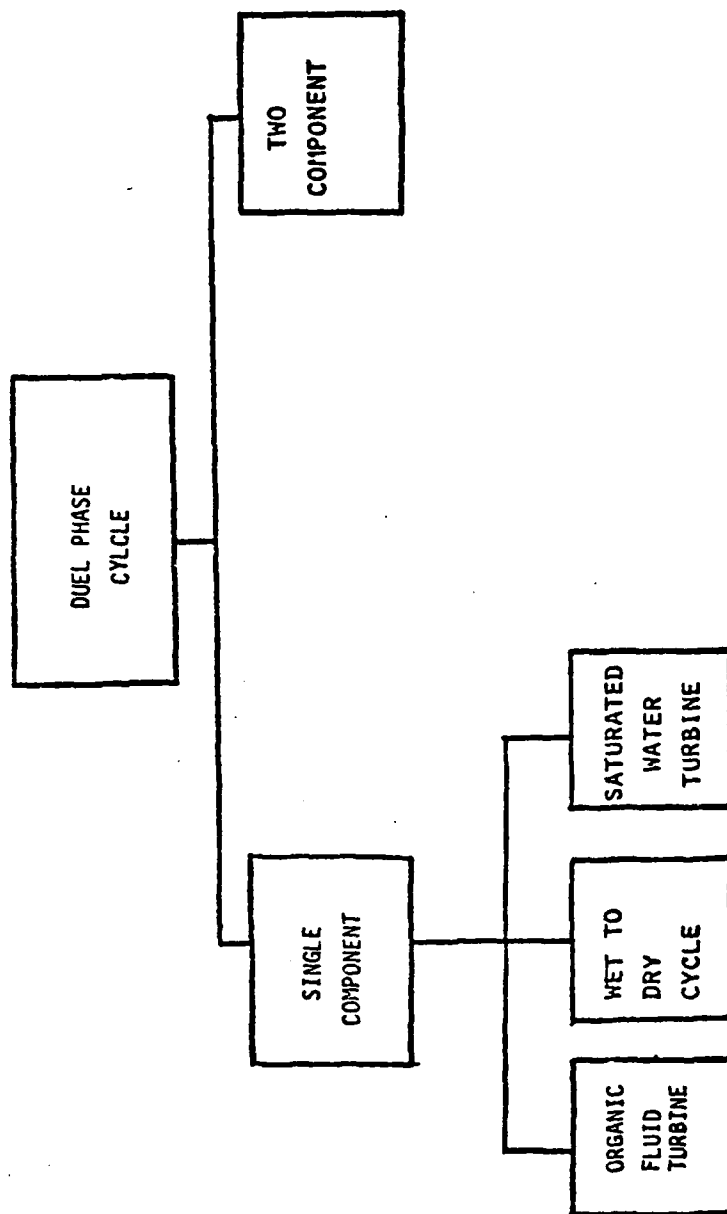


Figure 1. Division of the Dual-Phase Cycle

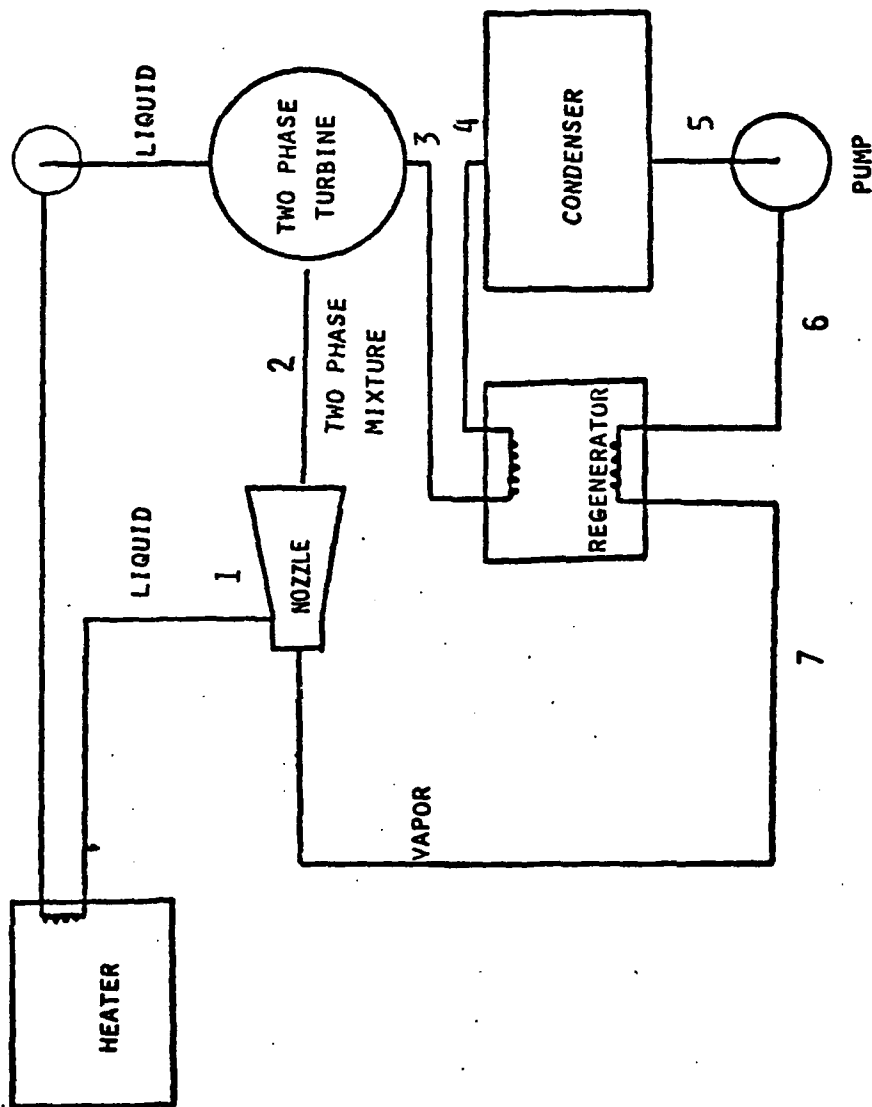


Figure 2. Dual-Phase Two-Component System

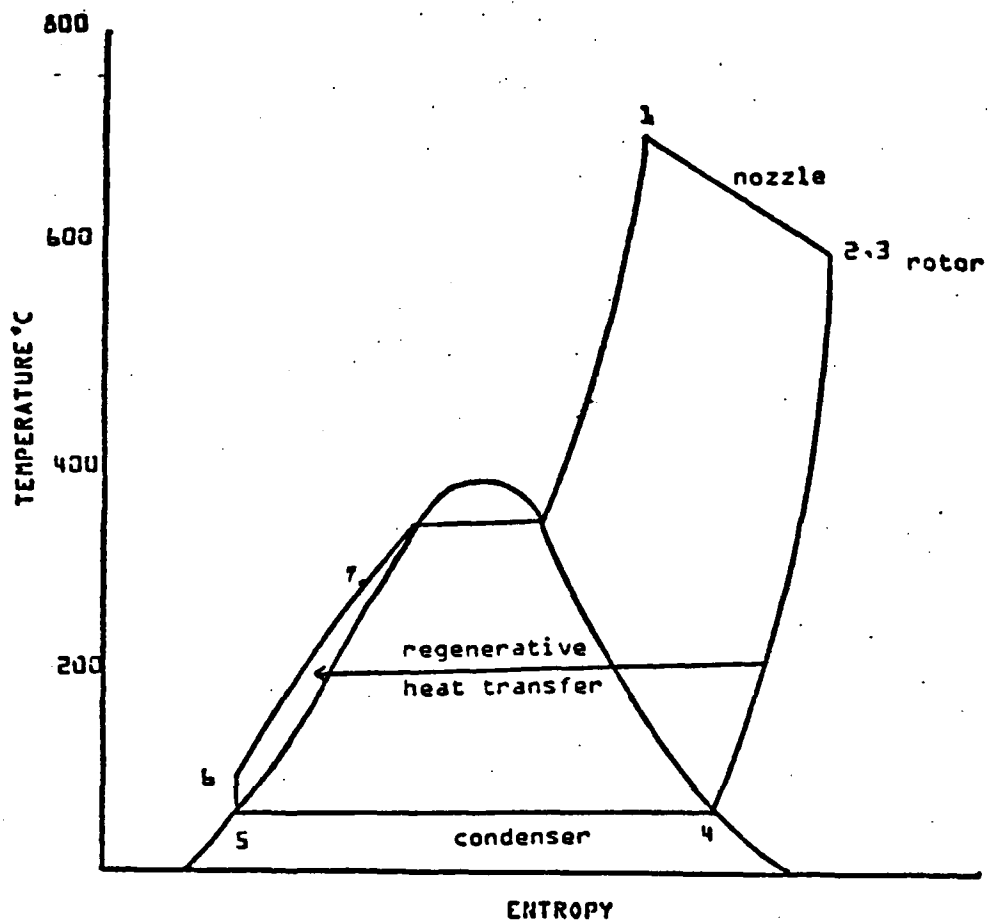


Figure 3. Dual-Phase Two-Component T-S Diagram

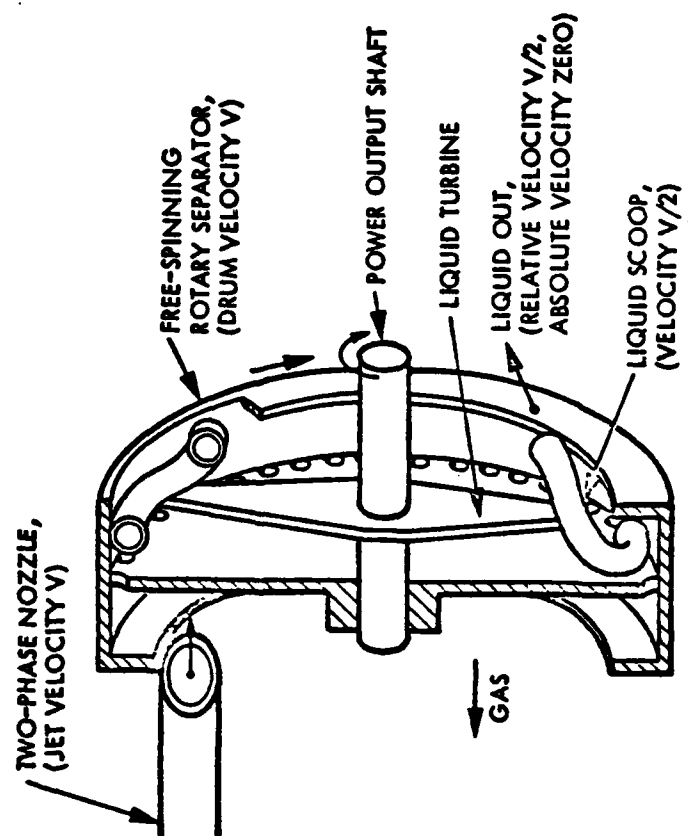
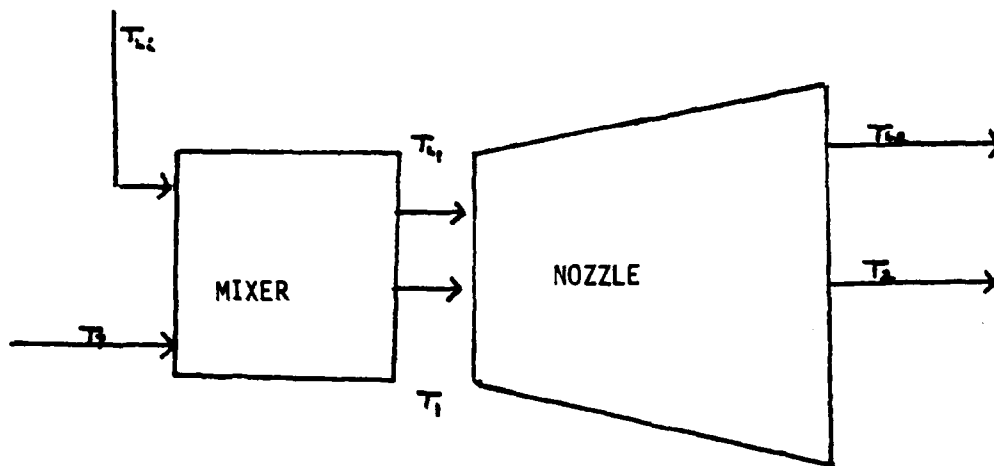


Figure 4. Liquid Impulse Turbine Schematic

expanded to a low pressure at the nozzle exit. The expanding vapor transfers momentum to the liquid droplets, while the droplets transfer heat to the vapor. This tends to approach isothermal expansion through the nozzle, point 2. After leaving the nozzle, the liquid droplets are separated from the vapor by a rotary separator located on the turbine. The rotating drum turns at close to the nozzle's exit velocity. Hence there is very little friction loss between the separated liquid and the drum wall. The centrifugal acceleration produces a very clean separation of vapor and liquid. The liquid traveling with the rim of the rotating drum transfers axially through holes in the separator disc and subsequently enters the liquid turbine proper. The kinetic energy of the liquid is converted to shaft horsepower by the liquid turbine. Figure 5 illustrates the dual-phase liquid impulse turbine assembly. The vapor remains superheated as a consequence of the heat transfer from the liquid. The vapor flows from the separator, point 3, through a regenerator where heat is transferred to the condensate. The vapor is condensed, point 5; pumped to nozzle pressure, point 6; and passed through the regenerator for heating. Heat is added between point 6 and point 1 in two methods. The regenerator adds heat to the condensate by using the steam from the turbine rotor, point 7. The remainder of heat, point 7 to 1, is added by the heated liquid mixed with the condensate in the mixer. The water is vaporized by direct-contact heat transfer.



where T_{L_i} = Temp of liquid entering mixer
 T_7 = Temp of water entering mixer
 $T_{L_i} > T_1$
 T_2 = Temp of steam leaving the nozzle
 T_{L_2} = Temp of liquid leaving the nozzle

Figure 5. Temperature & State Point Diagram for the Mixture & Nozzle

In the nozzle most of the thermal energy of the steam is converted to kinetic energy of the liquid droplets. This acceleration of the liquid by the vapor in the two-phase nozzle provides the kinetic energy to drive the liquid impulse turbine. The liquid velocities involved are relatively low as compared to velocity of the vapor. Thus, the output of the impulse hydraulic turbine will be high torque/low rpm. This conversion of the liquid kinetic energy to shaft power at high torque with low rpm appears to have direct application to naval propulsion.

B. ONE-COMPONENT

A one-component system is one in which the working fluid is of the same chemical compound. One of the simplest dual-phase one-component systems is illustrated in Figures 6 and 7. The working fluid is heated to saturation temperature by some type of heat source. Heat sources applicable to this case are geothermal power plants, engine exhaust, industrial waste-heat recovery, and bottoming cycles for steam and gas turbine plants. This working fluid, at saturated liquid conditions, with small amounts of vapor is expanded through a two-phase nozzle. As the expansion process takes place, the liquid partially vaporizes and accelerates the remaining liquid phase in the nozzle. The dual-phase mixture enters the rotary separator and the same process occurs as mentioned in section A. Since the liquid phase is of a much higher

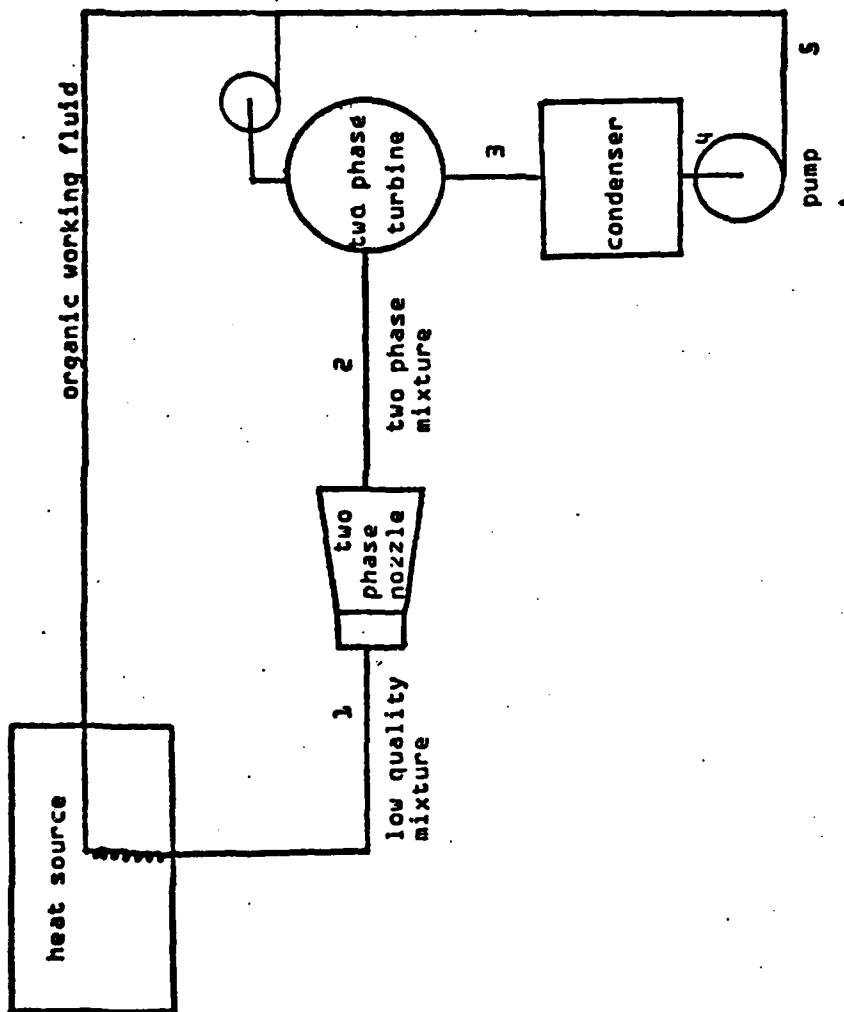


Figure 6. Dual-Phase Single-Component System

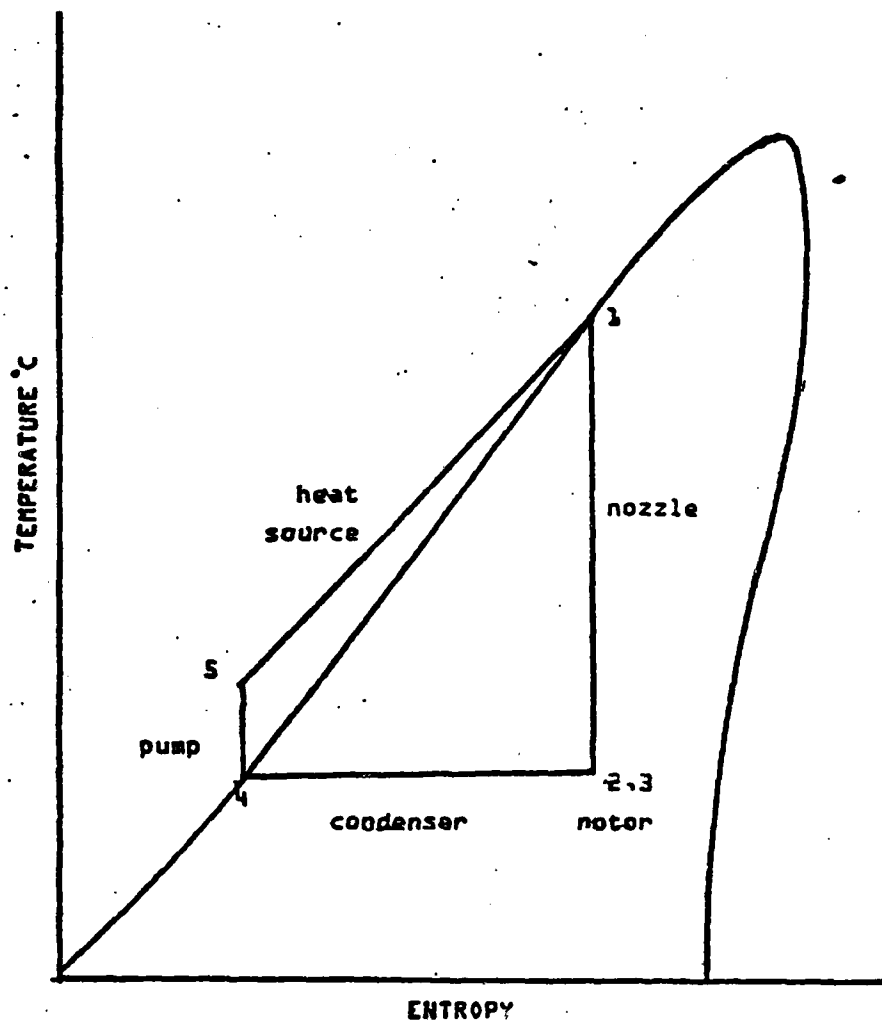


Figure 7. Dual-Phase Single-Component T-S Diagram

density than the vapor, the liquid velocities are relatively low as compared to the velocity of the vapor. Thus the output of the turbine will again have high torque at low rpm. After leaving the turbine the vapor mixture is condensed and the condensate is pumped back to the heat source. The cycle is shown on a T-S diagram in Figure 6. The state points are numbered to correspond with Figure 5. The dual-phase nozzle expansion takes the fluid from a saturated liquid, point 1, to a dual-phase flow, point 2. The flow is decelerated in the rotor; condensed, point 4; and pumped back to nozzle inlet pressure at point 5. The liquid is then reheated by the source fluid to point 1.

Another application of the one-component two-phase cycle is the wet-to-dry cycle. If the initial temperature of the working fluid is sufficiently high and the saturation curve has a positive saturated liquid slope the working fluid can be expanded to dry vapor. Figure 8 is the T-S diagram for a wet-to-dry cycle. The fluid is expanded from saturated liquid at point 1 to saturated vapor at point 2. The vapor drives an impulse rotor and leaves the rotor slightly superheated at point 3. The vapor is condensed to point 4 and pumped back to the nozzle inlet pressure at point 5.

C. ADVANTAGES

The advantage of the dual-phase cycle with respect to marine application is the ability to achieve low shaft speed

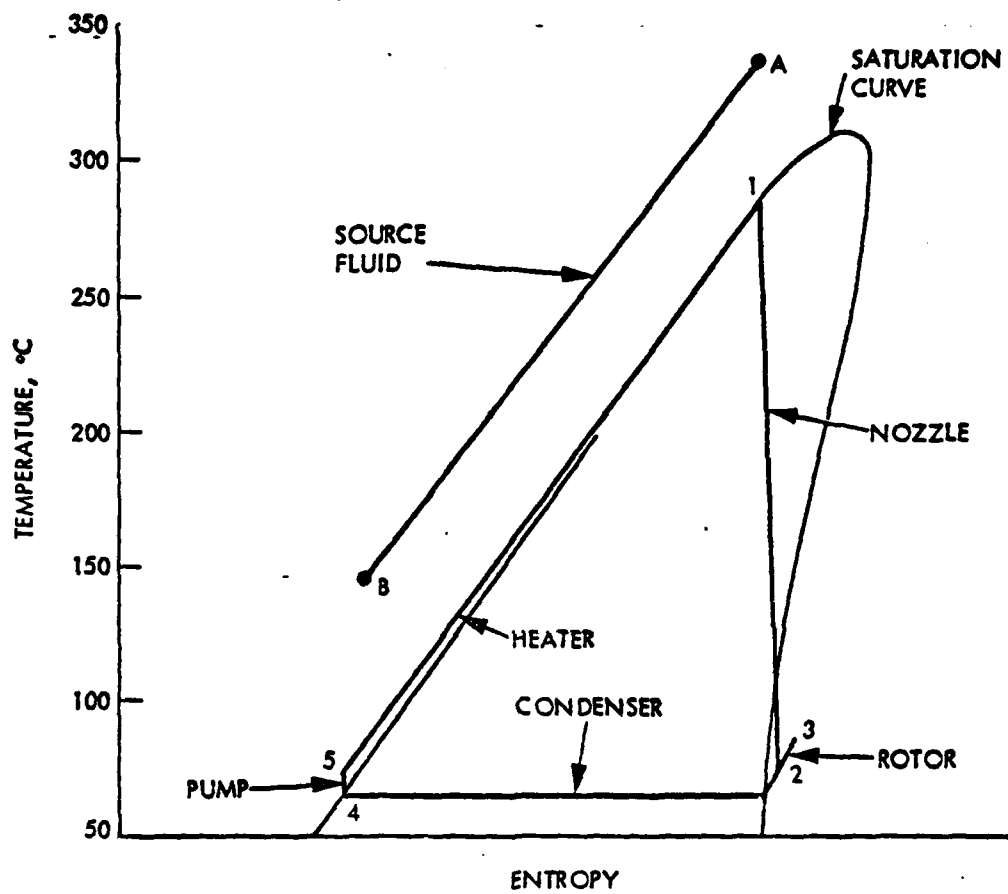


Figure 8. Wet-to-Dry T-S Diagram

in a small turbine engine. For example a steam turbine of 150-kw shaft horsepower using the temperature given in Figure 2 would have a speed of about 60,000 rpm. A comparable two-phase turbine would operate at approximately 10,000 rpm. There is also an efficiency advantage. At temperature corresponding to Figure 2, a steam Rankine cycle would have an efficiency of approximately 28% whereas the two-phase cycle would have an efficiency of 37%. This is assuming equal turbine efficiencies. The two-phase cycle also allows for control of turbine speed because the vapor/liquid mixture ratio can be varied to change the nozzle exit velocity. This is a capability unavailable in a conventional steam turbine.

Both of the dual-phase concepts can be thought of as a form of a regenerated Rankine cycle. The dual-phase cycle by control of liquid/vapor mixture ratio enhances the overall power system controllability. The T-S relationship for a dual-phase two-component engine cycle compared to a Rankine cycle is shown in Figure 9.

Two design studies References 1 and 2, have shown potential advantages in the two-phase engine cycle as compared to the conventional Rankine cycle for marine propulsion. The following advantages were noted:

1. High Efficiency - Full load output power performance gains ranging from 20 to 50 percent was found.
2. Direct Drive - Direct drive at speeds ranging from 90-4500 rpm was found possible with a single-stage turbine.

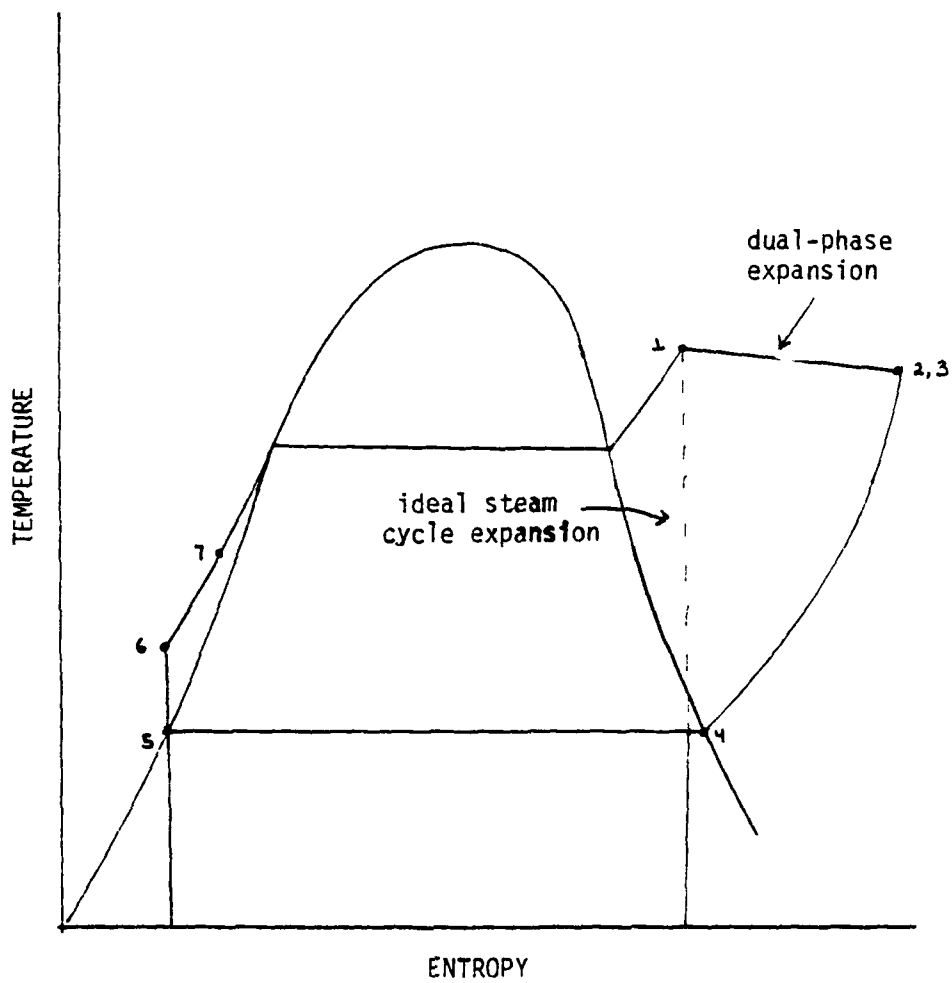


Figure 3. T-S Comparison of Dual-Phase Two-Component Cycle & Rankine Cycle

3. Reduced Volume - Volume reduction of 30 percent were estimated.

4. High Part-Load Efficiency - Variable mass ratio enabled part-load (cruise) efficiency gain of as much as 100 percent.

II. DUAL-PHASE NOZZLE THEORY

The flow phenomenon of a two-phase mixture has been analyzed in Reference 1. It is repeated as follows. The problem is illustrated in Figure 10. A spatially uniform two-component mixture of liquid drops and gas enters a nozzle at high pressure and low velocity and expands to low pressure and high velocity. The objective of the analysis is to determine, for a specified pressure the drop diameter D and the temperatures T_g and T_L , velocities V_g and V_L and flow rates $\dot{m}_g$ and $\dot{m}_L$ of the gas and liquid phases, respectively, at each station in the nozzle given the initial values of D , T_g , T_L , V_g , V_L , the total flow rate, and the properties of the fluids.

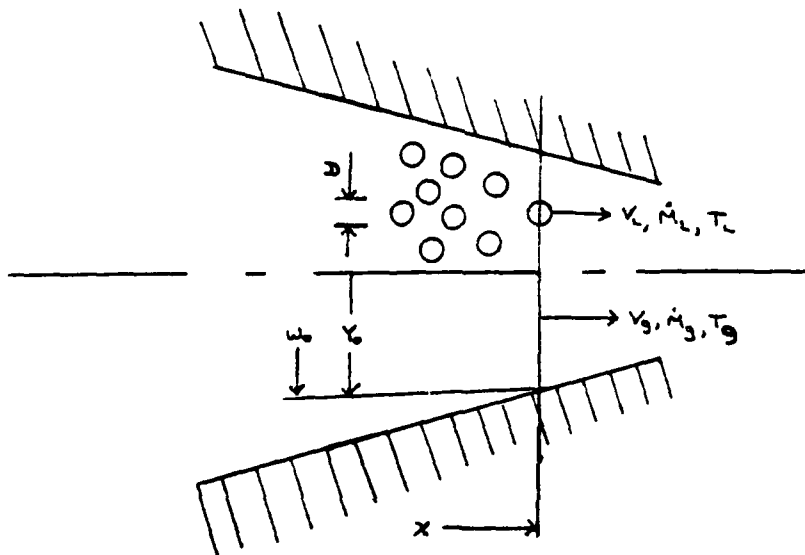


Figure 10. Dual-Phase Nozzle Flow Geometry & Nomenclature

The five relations employed to compute the five unknowns D , T_g , T_L , V_g , and V_L , are (1) the momentum equation for the mixture, (2) the energy equation for the mixture, (3) the drop drag equation, (4) the drop heat transfer equation, and (5) the drop breakup criterion. Solubility and vapor pressure relations provide the flow rate ratio $\dot{m}_g/\dot{m}_L$.

A. ASSUMPTIONS

The assumptions employed in the two-component analysis are as follows:

1. The liquid is uniformly dispersed as spherical drops all of the same diameter.
2. The drops break up to limit the Weber number to 6.
3. There are no external forces acting on the two-phase mixture other than pressure and wall shear, and there is no heat transfer to or from the mixture.
4. The flow is one-dimensional.
5. The drops are large enough for the surface curvature to have negligible effect on the vapor pressure of the liquid and for the surface energy to be negligible.
6. The drops are isothermal.
7. The gas mixture obeys the additive-pressure law.
8. The partial pressure of the predominantly liquid component in the liquid is given by Raoult's Law.
9. The concentration of the predominantly gaseous component in the liquid is given by Henry's Law.

10. The volume of the liquid solution is equal to the sum of the volumes of the pure liquids.

Assumption 1 restricts the analysis to nozzles having spatially uniform injection of the liquid into the gas and operating at gas-to-liquid volume ratios greater than unity. Assumption 2, the drop breakup criterion, states that drop diameter is limited to a value D for which $W_e = \rho_g V_s^2 D / 2\sigma = 6$. Thus

$$D_{\max} = \frac{12\sigma}{\rho_g V_s^2} \quad (1)$$

where ρ_g is the gas density, V_s is the slip velocity $V_g - V_L$, and σ is the liquid surface tension. The form of Eq. (1) is physically reasonable in that the Weber number is proportional to the ratio of stagnation pressure $\rho_g V_s^2 / 2$ to surface tension pressure $4\sigma/D$. Hence, a drop would be expected to flatten and breakup at a sufficiently high value of W_e . This has been verified experimentally and the critical Weber number found to be 6, within a factor of about two. An additional restriction is that for actual breakup to occur, the time spent at a Weber number exceeding 6 must be longer than the natural period of oscillation of the drop, $\pi(\rho_L D^3 / 4)^{1/2} / 4$, where ρ_L is the density of the liquid. This requirement is met only in two-phase nozzles longer than about 10 in. and Assumption 2 may cause the analysis to overestimate the exit velocity by increasing amounts as the nozzle length decreases below 10 in.

Assumption 3 excludes magnetohydrodynamic and mechanical body forces. The exclusion of wall heat transfer is correct for the insulated nozzles of interest for power systems. In addition, the relatively high velocity results in short residence lines in the nozzle proper.

Assumption 4 is closely met in practical nozzles since good performance requires small wall angles, large throat radius of curvature, and uniformly distributed injection of the fluids at the nozzle entrance.

Assumption 5 is valid for the drop sizes of 0.001 to 0.010 in. produced by the Eq. (1) breakup criterion. Assumption 6 is valid because of the rapid internal circulation in drops. Assumption 7 introduces negligible error in most cases of practical interest since the vapor pressure of the liquid is small and needs only to be evaluated approximately.

Assumptions 8, 9, and 10 are either valid, or cause little error, for fluids of low miscibility, which are the fluids of interest.

3. DERIVATION OF EQUATIONS FOR FREE-STREAM FLOW

1. Continuity

Referring to Figure 1, the nozzle flow area A is equal to the gas flow area $\dot{m}_g / \rho_g V_g$ plus the liquid flow area $\dot{m}_L / \rho_L V_L$. Thus

$$A = \dot{m}_g \left(\frac{1}{\rho_g V_g} + \frac{r}{\rho_L V_L} \right) \quad (2)$$

where r is the mass mixture ratio $\dot{m}_L / \dot{m}_g$.

2. Momentum

By Assumption 3, the only force acting on the free-stream flow is that due to the pressure gradient. If $\dot{M}$ is the momentum flux at flow area A, the change in momentum flux across pressure increment dp is

$$d\dot{M} = -A dp \quad (3)$$

The momentum flux can be written as the sum of the momentum fluxes of the gas and liquid. Thus,

$$\dot{M} = \dot{m}_g V_g + \dot{m}_l V_l \quad (4)$$

If the flow were allowed to continue at constant pressure, V_g and V_l would become equal to each other at the mass-weighted mean velocity $\bar{V}$. Since for this process, $d\dot{M} = 0$, the value of $\bar{V}$ is given by

$$(\dot{m}_g + \dot{m}_l)\bar{V} = \dot{m}_g V_g + \dot{m}_l V_l \quad (5)$$

or

$$\bar{V} = \frac{V_g + r V_l}{1 + r} \quad (6)$$

Thus, the momentum flux can be written

$$\dot{M} = (\dot{m}_g + \dot{m}_l)\bar{V} \quad (7)$$

Since $\dot{m}_g + \dot{m}_l$ is constant, the change in momentum flux is

$$d\dot{M} = (\dot{m}_g + \dot{m}_l)d\bar{V} \quad (8)$$

Substituting Eqs. (8) and (2) into Eq. (3), $d\bar{V}$ can be written

$$d\bar{V} = -\frac{1}{1+r} \left(\frac{1}{\rho_g V_g} + \frac{r}{\rho_l V_l} \right) dp \quad (9)$$

The slip ratio is defined as

$$s = V_s/\bar{V} = (V_s - V_L)/\bar{V} \quad (10)$$

This equation can be combined with Eq. (6) to give V_g and V_L in terms of $\bar{V}$:

$$V_s = \left(1 + \frac{rs}{1+r}\right) \bar{V} = a\bar{V} \quad (11)$$

$$V_L = \left(1 - \frac{s}{1+r}\right) \bar{V} = b\bar{V} \quad (12)$$

The gas density can be expressed as

$$\rho_g = W_g p / RT_g \quad (13)$$

where W_g is the effective molecular weight of the gas mixture and R is the universal gas constant. Eq. (13) is the definition of the effective molecular weight W_g , which is the quantity that gives the actual gas density when substituted in Eq. (13).

Substituting Eqs. (11) - (13) into Eq. (9), the differential momentum equation is

$$2\bar{V}d\bar{V} = d\bar{V}^2 = -\frac{2}{1+r} \left(\frac{RT_g}{aW_g p} + \frac{r}{b\rho_L} \right) dp \quad (14)$$

The quantities a and b are slowly varying because s is typically only 0.1 to 0.3 and slowly varying. The quantities r , T_g , W_g and ρ_L are also slowly varying. Integrating

Eq. (14) over a pressure increment Δp , for which a , b , r , T_g , W_g , and ρ_L are constant to within the desired accuracy, the change in $\bar{V}^2$ is

$$\Delta \bar{V}^2 = - \frac{2}{1+r_m} \left(\frac{RT_{g,m}}{a_m W_{g,m}} \log_e \frac{p + \Delta p/2}{p - \Delta p/2} + \frac{r_m \Delta p}{b_m \rho_{L,m}} \right) \quad (15)$$

All quantities other than pressure can be taken outside the integral and evaluated at their mean values (denoted by subscript m) corresponding to the mid-interval pressure p . Thus,

$$\begin{aligned} \Delta \bar{V}^2 = & - \frac{2}{1+r_m} \\ & \times \left(\frac{RT_{g,m}}{a_m W_{g,m}} \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} \frac{dp}{p} + \frac{r_m}{b_m \rho_{L,m}} \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} dp \right) \end{aligned} \quad (16)$$

Performing the integrations,

$$\Delta \bar{V}^2 = - \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} \frac{2}{1+r} \left(\frac{RT_g}{a W_{g,p}} + \frac{r}{b \rho_L} \right) dp \quad (17)$$

Equation (17) is the final form of the momentum equation.

3. Energy

The enthalpy change of the mixture between state 1 (the beginning of pressure interval Δp) and state 2 (the end

of the interval) can be evaluated in two steps: (1) phase change at p_1 , T_{g1} , T_{L1} and (2) change to p_2 , T_{g2} , T_{L2} , at fixed composition.

The enthalpy change for step 1 is

ΔH_1	=	amount of A vaporized	X	enthalpy required to vaporize and heat unit mass of A from T_L to T_{gL}
	+	amount of B vaporized	X	enthalpy required to vaporize and heat unit mass of B from T_{L1} to T_{g1}
	+	amount of A and B	X	kinetic energy required to accelerate unit mass from V_{L1} to V_{g1}

or

$$\begin{aligned} \Delta H_1 = & (\dot{m}_{a2} - \dot{m}_{a1}) [L_{a1} + c_{a1} (T_{g1} - T_{L1})] \\ & + (\dot{m}_{b2} - \dot{m}_{b1}) [L_{b1} + c_{b1} (T_{g1} - T_{L1})] \\ & + (\dot{m}_{g2} - \dot{m}_{g1}) (V_{g1}^2 - V_{L1}^2) / 2 \end{aligned} \quad (18)$$

where L and c are latent heat and specific heat, respectively.

Introducing more compact notation.

$$\begin{aligned} \Delta H_1 = & \Delta \dot{m}_a (L_{a1} + c_{a1} \delta_1 T) \\ & + \Delta \dot{m}_b (L_{b1} + c_{b1} \delta_1 T) + \frac{\Delta \dot{m}_g \delta_1 V^2}{2} \end{aligned} \quad (19)$$

The enthalpy change for step 2 is evaluated from the temperature, pressure, and velocity changes, with properties evaluated at mean T and p for the interval.

$$\begin{aligned}\Delta H_2 = & \dot{m}_{s_2} [c_{s_m} (T_{s_2} - T_{s_1}) + \frac{1}{2} (V_{s_2}^2 - V_{s_1}^2)] \\ & + \dot{m}_{i_2} [c_{i_m} (T_{i_2} - T_{i_1}) + \frac{p_2 - p_1}{\rho_{i_m}} \\ & + \frac{1}{2} (V_{i_2}^2 - V_{i_1}^2)]\end{aligned}\quad (20)$$

$$\begin{aligned}= & \dot{m}_{s_2} \left(c_{s_m} \Delta T_s + \frac{\Delta V_s^2}{2} \right) \\ & + \dot{m}_{i_2} \left(c_{i_m} \Delta T_i + \frac{\Delta p}{\rho_{i_m}} + \frac{\Delta V_i^2}{2} \right)\end{aligned}\quad (21)$$

By Assumption 3, no work is done by the free-stream flow and no heat is transferred to it. Hence,

$$\Delta H_1 + \Delta H_2 = 0 \quad (22)$$

Substituting Eqs. (19) and (21) into Eq. (22) and solving for ΔT_g gives the energy equation for the mixture:

$$\begin{aligned}\Delta T_g = & -\frac{1}{c_{g_m}} \left[\frac{\Delta V_g^2}{2} + r_2 \left(c_{i_m} \Delta T_i + \frac{\Delta p}{\rho_{i_m}} + \frac{\Delta V_i^2}{2} \right) \right. \\ & + \frac{\Delta \dot{m}_s \delta_1 V_s^2}{2 \dot{m}_{s_2}} + \frac{\Delta \dot{m}_{s_2}}{\dot{m}_{s_2}} (L_{s_1} + c_{s_g} \delta_1 T) \\ & \left. + \frac{\Delta \dot{m}_{i_2}}{\dot{m}_{s_2}} (L_{i_1} + c_{i_g} \delta_1 T) \right]\end{aligned}\quad (23)$$

4. Drag

Although no force other than pressure acts on the free-stream flow as a whole, a drag force exists between

the phases. Hence, a second momentum equation must be written using as the control volume the boundary between the phases.

The two forces acting on each liquid drop are the buoyancy due to the pressure gradient and the drag due to the relative gas velocity. The sum of these is equal to the mass times the acceleration of the drop. Thus, for a single drop.

$$\begin{array}{rcl}
 \text{dynamic pressure of} & & \text{drag} \\
 \text{relative gas flow} & \times & \text{coefficient} \times \\
 & & \text{frontal area} \\
 & & \text{of drop} \\
 \\
 - \text{volume} & \times & \text{pressure} \\
 \text{of drop} & & \text{gradient} \\
 & = & \text{mass} \\
 & & \text{drop} \times \text{acceleration} \\
 & & \text{of drop}
 \end{array}$$

or

$$\begin{aligned}
 \left(\frac{1}{2} \rho_g |V_g| V_g \right) C_D \frac{\pi D^2}{4} - \frac{\pi D^3}{6} \frac{dp}{dx} \\
 = \left(\frac{\pi D^3}{6} \rho_l \right) \left(V_L \frac{dV_L}{dx} \right)
 \end{aligned} \tag{24}$$

The absolute value sign in the first term makes the drag force positive when $V_g > V_L$ and negative when $V_g < V_L$.

Solving Eq. (24) for dV_L ,

$$dV_L = \frac{3\rho_g |V_g| V_g C_D dx}{4\rho_l V_L D} - \frac{dp}{\rho_l V_L} \tag{25}$$

Differentiating Eq. (12), dV_L can also be expressed in terms of s , r , and $\bar{V}$. Thus,

$$dV_L = b d\bar{V} + \bar{V} \left[\frac{s dr}{(1+r)^2} - \frac{ds}{1+r} \right] \quad (26)$$

Solving for ds ,

$$ds = \frac{b(1+r) d\bar{V}}{\bar{V}} + \frac{s dr}{1+r} - \frac{(1+r) dV_L}{\bar{V}} \quad (27)$$

Substituting dV_L from Eq. (25), noting that $d\bar{V} = d\bar{V}^2/2\bar{V}$, using Eq. (12), and writing for a finite increment, results in,

$$\Delta s = \frac{b_m(1+r_m) \Delta \bar{V}^2}{2\bar{V}_m^2} + \frac{(1+r_m) \Delta p}{b_m \rho_{L_m} \bar{V}_m^2} + \frac{s_m \Delta r}{1+r_m} - \frac{3\rho_{r_m} |s_m| s_m C_{D_m} (1+r_m) \Delta x}{4 b_m \rho_{L_m} D} \quad (28)$$

This is the drag equation employed when x is specified as a function of p .

Solving Eq. (28) for Δx yields the required alternative equation:

$$\Delta x = \frac{4D}{3\rho_{r_m} |s_m| s_m C_{D_m} \bar{V}_m^2} \left[\Delta p + \frac{b_m^2 \rho_{L_m} \Delta \bar{V}^2}{2} + \frac{b_m \rho_{L_m} \bar{V}_m^2}{1+r_m} \left(\frac{s_m \Delta r}{1+r_m} - \Delta s \right) \right] \quad (29)$$

5. Heat Transfer

Although no heat is transferred to the mixture as a whole, heat transfer exists between the phases. Hence, a second energy equation must be written using as the control volume the boundary between the phases.

The work dW done on the liquid is that due to drag by the gas. (Only work done by shear or shaft forces is included in dW when writing the First Law for a control volume). Multiplying Eq. (24) by the number flow rate of drops $\dot{N} = 6\dot{m}_L/\pi D^3 \rho_L$, the drag force F_d on that quantity of liquid is

$$F_d = \frac{\dot{N}}{8} \rho_g |V_r| V_r C_{D0} D^2 = \frac{\dot{m}_1}{\rho_1} \frac{dp}{dx} + \dot{m}_1 V_1 \frac{dV_1}{dx} \quad (30)$$

The work done on the liquid is

$$-dW = F_d dx = \dot{m}_1 \left(\frac{dp}{\rho_1} + \frac{dV_1^2}{2} \right) \quad (31)$$

The heat dQ transferred from the liquid is made up of two parts: (1) the convective cooling due to the temperature difference between the liquid and gas and (2) the evaporative cooling due to the latent heat supplied to the liquid vaporized. The convective cooling is

$$-dQ_c = h A_s \dot{N} (T_1 - T_g) dt \quad (32)$$

where h is the heat-transfer coefficient, $A_d = \pi D^2$ is the surface area of a drop, and $dt = dx/V_L$ is the time required to traverse dx . Thus,

$$-dQ_c = \frac{6h\dot{m}_l(T_g - T_l)dx}{D\rho_l V_l} \quad (33)$$

The evaporative cooling is

$$-dQ_e = L_e d\dot{m}_e + L_v d\dot{m}_v, \quad (34)$$

The change in enthalpy of the liquid over the pressure increment dp is

$$dH = \dot{m}_l \left(c_l dT_l + \frac{dp}{\rho_l} + \frac{dV_l^2}{2} \right) \quad (35)$$

Substituting Eqs. (31), (33), (34), and (35) into the steady-flow energy equation $dQ - dW = dH$, the result is

$$\frac{6h\dot{m}_l \delta T dx}{D\rho_l V_l} - L_e d\dot{m}_e - L_v d\dot{m}_v = \dot{m}_l c_l dT_l \quad (36)$$

where $\delta T = T_g - T_l$.

Writing for a finite interval, the final form of the drop heat-transfer equation is

$$\Delta T_l = \frac{1}{c_{l_m}} \left[\frac{6h\delta_m T \Delta x}{D\rho_{l_m} V_{l_m}} - L_{e_m} \frac{\Delta \dot{m}_{e_l}}{\dot{m}_{l_m}} - L_{v_m} \frac{\Delta \dot{m}_{v_l}}{\dot{m}_{l_m}} \right] \quad (37)$$

Equations (1), (17), (23), (23), and (37) are the five equations that must be solved simultaneously to obtain the values of the five dependent variables D , T_g , T_L , V_g , and V_L as a function of the independent variable p . To carry out the solution all quantities in the equations must be expressed in terms of these six variables.

C. WALL SHEAR AND BOUNDARY LAYER

For a two-phase nozzle, the momentum flux of the frictionless nozzle flow is that given by

$$\dot{M} = \dot{m}_t \bar{V}$$

The mean mixture density corresponding to the mean velocity $\bar{V}$ is

$$\rho' = \frac{\dot{m}_t}{A\bar{V}} = \frac{\rho_l}{1 + r_a} \quad (38)$$

where r_a is the ratio of gas flow area to liquid flow area $\rho_l V_l / \rho_g V_g$.

From the definition of the momentum thickness, the value of θ at a station where the nozzle wall radius is y_0 is given by

$$\dot{M} - \dot{M}_f = 2\pi y_0 \theta \frac{\dot{m}_t \bar{V}}{A} = 2\pi y_0 \rho' \bar{V}^2 \theta \quad (39)$$

where $\dot{M}_f$ is the momentum flux of the real flow with friction.

The skin-friction coefficient can be defined using the same quantities as single-phase flow

$$C_f = \frac{2\tau_w}{\rho \bar{V}^2} \quad (40)$$

where τ_w is the wall shear. It will be shown that a valid C_f value can be provided.

The boundary-layer momentum equation then becomes

$$d\theta = \frac{C_f}{2} dx - \theta \left[\frac{1 + (\delta^*/\theta)}{\bar{V}} d\bar{V} + \frac{1}{\rho \bar{V}} d(\rho \bar{V}) + \frac{1}{R_w} dR_w \right] \quad (41)$$

where δ^* is the displacement thickness, i.e., the distance the wall must be moved outward to give the same flow rate as with frictionless flow.

Assuming a $\frac{1}{2}$ power velocity profile and no density variation, the shape factor δ^*/θ is obtained from

$$\frac{\delta^*}{\theta} = \frac{\int_0^\delta [1 - (y/\delta)^{1/2}] dy}{\int_0^\delta (y/\delta)^{1/2} [1 - (y/\delta)^{1/2}] dy} = \frac{9}{7}$$

where δ is the velocity thickness of the boundary layer.

Noting that $d\bar{V}$ can be written $d\bar{V}^2/2\bar{V}$, and that $d(\rho \bar{V}) = d(m_t/A)$, the finite-difference form of Eq. (41) is

$$\Delta\theta = \frac{C_{f_m}}{2} \Delta x - \theta_m \left(\frac{8}{7\bar{V}_m} \Delta\bar{V}^2 - \frac{1}{A_m} \Delta A + \frac{1}{y_{om}} \Delta y_o \right)$$

Wall shear in homogeneous two-phase flow has been found to be equal to that which would exist with pure liquid at equal velocity and boundary-layer thickness, multiplied by the wetted wall fraction:

$$\tau_w = \frac{C_{f_l}}{2} \rho_l V_l^2 \frac{A_l}{A} = \frac{C_{f_l} \rho_l V_l^2}{2(1+r_a)} \quad (42)$$

where C_{f_l} is the skin friction coefficient for liquid at a Reynolds number of

$$R_\delta = \frac{\rho_l V_l \delta}{\mu_l}$$

For a 1/7-power profile, the velocity thickness,

$$\delta = \frac{72}{7} \theta$$

A convenient relation for C_{f_l} as a function of R_δ is the Shultz-Grunow relation which can be written

$$C_{f_l} = \frac{0.208}{(\log_{10} R_\delta + 0.425)^{2.584}}$$

Comparison of Eqs. (40) and (42) shows that C_f can be written

$$C_f = \frac{rb}{1+r} C_{f_l}$$

Thus, the final form of the boundary-layer momentum equation is

$$\Delta\theta = \frac{r_m b_m}{1+r_m} \frac{C_{f_l} \Delta x}{2} - \theta_m \left(\frac{8\Delta\bar{V}^2}{7\bar{V}_m^2} - \frac{\Delta A}{A_m} + \frac{\Delta y_n}{y_{on}} \right)$$

Let $\bar{V}_\delta$ be the mean velocity of the flow including the boundary layer. Then, from Eq. (39),

$$\dot{m}_i \bar{V}_\delta = \dot{M}_i = \dot{m}_i \bar{V} - 2\pi y_o \rho \bar{V}^2 \theta$$

Hence, employing Eq. (38), the mean exit velocity including the boundary layer is

$$\bar{V}_\delta = \bar{V} \left(1 - \frac{2\pi y_o \theta}{A} \right)$$

By the definition of the displacement thickness, the flow rate is reduced by the throat displacement thickness δ_t^* to

$$\dot{m}_\delta = \dot{m}_i \left(1 - \frac{2\delta_t^*}{y_o} \right)$$

D. NOZZLE THEORY SUMMARY

The preceding equations form the basis for the mathematical model which is used to predict, based on inlet conditions, the exit velocity, and temperature of the mixture. These equations also form the basis for the model which provides the optimum nozzle shape given a set of inlet conditions. Some additional relationships are, however, required. These are:

1. Phase properties - to establish the mass ratio, mass flow rate ratio of gas to liquid, and the thermal conductivity of the mixture.

2. Liquid drop drag coefficients.
3. The liquid drop heat transfer coefficients.
4. Boundary layer momentum thickness and displacement thickness.
5. Skin friction coefficient.

These five additional relationships are developed in detail in Reference [1].

III. COMPUTER PROGRAM DUAL-PHASE NOZZLE

The computer program employed in this study is based on a program developed by Dr. G. Elliott of the Jet Propulsion Laboratory in Pasadena, California. The program was updated and converted for use on the Naval Postgraduate School computer. The Dual-Phase Two-Component program employs the theory in Section II. The program is written in Fortran computer language and can be compiled using a Watfiv or Fortran IV compiler.

This program has been utilized in dual-phase nozzle analysis and to provide values for comparison with the experimental results. To use the computer program the inlet conditions have to be specified. The flow conditions are: inlet pressure; mass ratio; inlet temperature of the gas and liquid; inlet velocity of the gas and liquid; total mass flow rate; and nozzle exit pressure. Section III B, shows specified details for data input.

There are two options that can be chosen. The first is prescribed pressure-versus-distance option MOP=0. The pressure profile $P(X)$ is selected corresponding to the adopted nozzle contour. If the pressure-versus-distance is used a $P(X)$ input table is required. This profile is developed from the actual measured pressure values in the experiments.

(See Appendix A for sample program.) The second option consist of an optimum nozzle contour option MOP=1. This option is useful only when the liquid drop diameter is constant.

The dual-phase two-component computer program is a structured program with thirteen subroutines controlled by a main program. This arrangement improved the programming process through better organization and programming notation.

The control point of the dual-phase two-component computer program is the "main section." It controls the flow path and operation of all input data, property tables, and calculations. It accomplishes this by calling the thirteen subroutines at the appropriate times, saving wanted data in files, and printing out desired information.

One of the most important subroutines which inputs information is the "INTRP" subroutines. INTRP controls the property table inputs. It reads in four two-dimensional tables and fourteen one-dimensional tables. These inputs are the properties of the gas and liquid phases of both components of the flow in the nozzle. The subroutine writes the values of these tables into a file and retrieves appropriate values from that file. INTRP can also interpolate for values used throughout the entire program.

Input of case data is controlled by subroutine "Sect 1." Identification information and case heading information is read and printed for each instance. Sect 1 also places the pressure vs. distance profile, if specified, in an array.

"Sect 2" through "Sect 6" are the subroutines which calculate the flow data. Sect 2 sets the initial conditions indicated in the input, and begins the iterations. Sect 3 computes initial flow rates of both components; the initial area of the nozzle; slip velocity; mean free stream velocity; and slip friction. Sect 4 computes the changes in flow parameters and new distances. It then begins to calculate new conditions such as flow rate, temperature, velocities, surface tension and mean area. Sect 5 is the binary cut convergence routine and computes mean boundary-layer parameters.

If a problem is diagnosed in any subroutines and "diagnose" is called, it will print all output parameters calculated to that point. It also does the same if there is a convergence problem.

There are two subroutines that output calculated data, subroutine "Write" and subroutine "Output." Subroutine "Write" will send output information to the printer for a hard paper copy. Subroutine "Output" reads and stores the output on a file.

The two-phase two-component program has been written with comment statements in the text of the program. These will allow for a more understandable and, therefore, a more easily modified program. For specific details on the content of these subroutines, see Appendix M.

The dual-phase nozzle program was tested for correct output. Sample data and results were obtained from

Dr. David Elliott, were inputted into the program, and executed. The output was compared to the sample data. The program produced duplicate results.

The program begins by storing fluid property tables and reading in all input data. All nozzle inlet conditions are computed. The program then proceeds half a pressure step at a time. At the middle of each pressure interval, the changes in quantities across the interval are computed using the properties interpolated from the table for that pressure, and for the existing temperature. The change in slip is found if the pressure profile $P(X)$ is specified. At the end of each pressure step, the flow conditions are updated and initial conditions are determined for the next step. The dropsize is reduced at the point when the Weber number exceeds six. The flow conditions are printed if the pressure is one selected for output. The computation continues until the last pressure step has been completed and flow conditions at the smallest flow area encountered are printed as the throat conditions.

A. PROPERTY TABLES

1. Heat capacity of component "A" vapor in BTU/LBM-R is a function of temperature and pressure. The two-dimensional tables are entered row-wise. At least two cards are necessary to specify a row and at least two rows must be entered.

Card 1: (format 6E12.6)

cols.

1-12 temperature (R)

13-24=1.0 if this is the last temp for
this table

Card 2: (format 6E12.6)

cols.

1-12 pressure (psi)

13-24 heat capacity (BTU/LBM-R)

25-36 pressure (psi)

37-48 heat capacity (BTU/LBM-R)

49-60 pressure (psi)

61-72 heat capacity (BTU/LBM-R)

The maximum entries of temperature are 35 values. For each value of temperature, the maximum number of entries of pressure and heat capacities are 35 values. Each row of this table will be terminated with the pressure and heat capacity equal to 10^5 . These two values are not counted in the maximum of 35 entries/rows allowed.

The program shown in Appendix B can be used to determine the values of heat capacities. The program structures its output in the format needed for the table input. It uses input data obtained from Reference 3. The input data must be placed in a two-dimensional table. This table is used in the program to interpolate the values needed for output. Input data must be formatted as follows:

Card 1-16 (format 10F7.4)

cols.

1-7 temperature (R)

at this temp the following is entered:

8-14 heat capacity at .01 P

15-21 heat capacity at .4 Pa

22-28 heat capacity at .7 Pa

29-35 heat capacity at 1.0 Pa

34-42 heat capacity at 4.0 Pa

43-49 heat capacity at 7.0 Pa

50-56 heat capacity at 10.0 Pa

57-63 heat capacity at 40.0 Pa

64-70 heat capacity at 70.0 Pa

The temperature must be entered with
increasing value.

2. Heat capacity of component "B" gas, BTU/LBM-R is a function of temperature and pressure. This two-dimensional table has the same format as part A1 above.

3. Molecular weight of component "A" vapor is a function of temperature and pressure. The two-dimensional tables are entered row-wise. At least two cards are necessary to specify a row and at least two rows must be entered.

Card 1: (format 6E12.6)

cols.

1-12 temperature (R)

13-24=1.0 if this is the last temp for
this table

Card 2: (format 6E12.6)

cols.

1-12 pressure (psi)

13-24 molecular weight

25-36 pressure (psi)

37-48 molecular weight

49-60 pressure (psi)

61-72 molecular weight

The maximum number entries of temperature are 35 values. For each value of temperature, the maximum entries of pressure and molecular weight are 35 values. Each row of this table will be terminated with the pressure and molecular weight equal to 10^5 . These two values are not counted in the maximum of 35 entries/rows allowed.

The program shown in Appendix C can be used to determine the values of molecular weight. The program formats its output in the format needed for the table input of the two-component two-phase computer program. It uses input data obtained from Reference 3. The input data must be placed in a two-dimensional table. This table is used in the program to interpolate the values needed for output. The program used in Figure 8 can only be used with ideal gases. Input data must be formatted as follows:

Card 1-16 (format 10F7.4)

cols.

1-7 temperature (R)

at this temp the following is entered:

8-14 density at .01 P

15-21 density at .4 Pa

22-28 density at .7 Pa

29-35 density at 1.0 Pa

34-42 density at 4.0 Pa

43-49 density at 7.0 Pa

50-56 density at 10.1 Pa

57-63 density at 40.0 Pa

64-70 density at 70.0 Pa

The temperatures must be entered with
increasing value.

4. Molecular weight of component "B" gas is a function of temperature and pressure. This two-dimensional table has the same format as part A3 above.

5. There are fourteen one-dimensional tables. The one-dimensional tables are entered in the following format (for Z(T)):

Card 1: (format 6E12.6)

cols.

1-12 Ti-2, oR

13-24 Ti-2

25-36 T_i-1 $2 \leq i \leq 50$

37-48 Z_i-01

49-60 T_i

61-72 Z_i and $T_j-1 < T_j$ for $2 \leq j \leq 50$

Each table is terminated by two consecutive entries of 10^5 , i.e., $T_k \equiv Z_k \equiv 10^5$ (1.0E5 right adjusted in the field).

The fourteen one-dimensional tables are (in order requested):

1. CAL(T) heat capacity of component A liquid,
BTU/LBM or
2. CBL(T) heat capacity of component B liquid,
BTU/LBM or
3. LA(T) latent heat of vaporization for com-
ponent A, BTU/LBM
4. LB(T) latent heat of vaporization for com-
ponent B, BTU/LBM
5. PBO(T) vapor pressure of component B
6. ROAL(T) density of liquid component A,
LBM/FT³
7. ROBL(T) density of liquid component B,
LBM/FT³
3. KAG(T) thermal conductivity of component
A gas, BTU/FT HR or
9. KBG(T) thermal conductivity of component
B gas, BTU/FT HR or

10. VIAL(T) viscosity of liquid component A,
LBM/FT HR
11. VIBL(T) viscosity of liquid component B,
LBM/FT HR
12. VIAG(T) viscosity of gas component A,
LBM/FT HR
13. VIBG(T) viscosity of gas component B,
LBM/FT HR
14. SIG(T) surface tension of liquid component
B, DYNE/CM

Appendix D gives sample property table for input.

B. CASE INPUT

A blank card must separate the property table from the data set decks following.

Card 1: (format 4A4, A2, 3A4, 7A4, A32, 2I6)
cols.

1-18 date

19-30 case number, may be any

alphanumeric data

31-60 identification

61-66 NS, number of pressure steps per
printout

(right justified integer).* Use -1 if new
property tables follows.

67-72 NP, number of printouts (right
justified)

Card 2: (All integers right justified in field)

(format 1116)

cols.

1-6 MBU, =0 constant droplet size, =1 drop
breakup

7-12 MOP, =0 X(P) table to be supplied, =1
X(P) is to be optimized

13-18 MGEO, =0 circular, =1 annular

19-24 NDS, maximum number of S iterations

25-30 NSO, maximum number of So iterations

31-36 NB, maximum number of TB iterations

37-42 NNS, first setting of step counter

Card 3: (format 5E12.6)

cols.

1-12 DP, pressure step size, negative for
decreasing, psi

13-24 RC, mass flow ratio

25-36 PHI, angle of annular nozzle axis, deg

37-48 RAXO, annular nozzle axis, in.

49-60 EMT, total flow rate *P final = $P_0 +$
 $(NS*NP-NNS-1)*DP+DP1$

Card 4: (format 6E12.6)

cols.

1-12 H, inverse Henry's Law constant, psi

13-24 ALAM, Lagrangian multiplier

25-26 DPL, first pressure step size
37-48 WAL, molecular weight of liquid a
49-60 WBL, molecular weight of liquid b
61-72 SA, Sutherland constant for component
A, oR

Card 5: (format 6E12.6)

cols.

1-12 SB, Sutherland constant for component
B, oR
13-24 DO, initial drop diameter, in.
25-36 PO, initial pressure, psia
37-48 TGO, initial gas temperature, oR
49-60 TLO, initial liquid tempeature, oR
61-72 VGO, initial gas velocity, FT/S

Card 6: (format 6E12.6)

cols.

1-12 VLO, Initial liquid velocity, FT/S
13-24 THOO, initial momentum thickness of
outer wall boundary layer, in.
25-36 THIO, initial momentum thickness of
inner wall boundary layer, in.
37-48 EDS, convergence criterion for S
49-60 ESO, convergence criterion for So
61-72 EB, convergence criterion for T, oR
If MOP=0, the following cards are present:
in 10 Card 7: (format 7A4, A2) cols.

1-30 X(P) table identification (and
alphanumeric data).

Card 8: (format 6E12.6)

cols.

1-12 pressure, pi-2, psia

13-24 distance, xi-2, in.

25-36 pressure, pi-1 $3 \leq i \leq 75$

37-48 distance, xi-1

49-60 pressure, pi

61-72 distance, xi

The last two entries are 1.0E5 and 1.0E5 right adjusted in their fields. The table must be monotonic increasing or decreasing. New property tables may be used by putting -1 in cols. 61-66 of Card 1, and following this with new property tables and data sets. Appendix E is a sample input data.

C. OUTPUT

For each case, the case identification is printed followed by the input parameters. If MOP=0, the X(P) table forms a part of this output. The following output then appears.

1. X distance, in.
2. P pressure, psia
3. R mass flow ratio
4. vb mean free-stream velocity, ft/s
5. a flow area, in. ²

- 6. tb gas temperature, or
- 7. tl liquid temperature, or
- 8. vg gas velocity, ft/s
- 9. vl liquid velocity, ft/s
- 10. vs slip velocity $vg - vl$, ft/s
- 11. s slip fraction vs/vb
- 12. d drop diameter, in.
- 13. rv ratio of gas volume flow to liquid volume flow
- 14. ra ratio of gas flow area to liquid flow area
- 15. alpha mass fraction of component a dissolved in liquid
- 16. beta mass fraction of component b vapor in gas
- 17. mg gas flow rate, lbm/s
- 18. ml liquid flow rate, lbm/s
- 19. rog gas density, lbm/ft³
- 20. rol liquid density, lbm/ft³
- 21. wag molecular weight of component a gas
- 22. wbg molecular weight of component b gas
- 23. wg mean molecular weight of gas
- 24. pa partial pressure of component a, psia
- 25. pb partial pressure of component b, psia
- 26. la latent heat of vaporization of component a, btu/lbm
- 27. lb latent heat of vaporization of component b, btu/lbm

28. sigma liquid surface tension, dyne/cm
29. cgm specific heat of gas (at midpoint of pressure step), btu/lbm of
30. clm specific heat of liquid, btu/lbm of
31. vigm viscosity of liquid, lbm/ft hr
32. vilm viscosity of gas, lbm/ft hr
33. kgm thermal conductivity of gas, btu/hr ft of
34. rem reynolds number of flow over drops
35. cdm drag coefficient of drops
36. hm heat transfer coefficient of drops, btu/hr ft²
37. rb mass flow ratio after velocity and temperature equalization
38. ab flow area after equalization, in. ²
39. tb temperature after equalization, or
40. rvb volume flow ratio after equalization
41. alphb alpha after equalization
42. betab beta after equalization
43. mgb gas flow rate after equalization, lbm/s
44. mlb liquid flow rate after equalization, lbm/s
45. vilb liquid viscosity after equalization, lbm/ft hr
46. rogb gas density after equalization, lbm/ft³
47. rolb liquid density after equalization, lbm/ft³
48. wagb molecular weight of component a gas after equalization

49. wbgb molecular weight of component b gas after
 equalization
50. wgb mean molecular weight of gas after equalization
51. pab partial pressure of component a gas after
 equalization, psia
52. pbb partial pressure of component b gas after
 equalization, psia
53. g separator friction parameter
54. ref separator film reynolds number
55. yo distance from nozzle axis to outer wall, in.
56. wom angle of outer wall relative to axis, deg
57. tho momentum thickness of outer boundary layer, in.
58. delo velocity thickness of outer boundary layer, in.
59. delso displacement thickness of outer boundary layer,
 in.
60. redom reynolds number of outer boundary layer
61. cfom skin friction coefficient of outer boundary
 layer
62. twom shear stress on outer wall, psi
63. vbd mean velocity including boundary layer, ft/s
64. wim angle of inner wall relative to axis, deg
65. thi momentum thickness of inner boundary layer, in.
66. deli velocity thickness of inner boundary layer, in.
67. delsi displacement thickness of inner boundary
 layer, in.

68. redim reynolds number of inner boundary layer
69. cfim skin friction coefficient of inner boundary
layer
70. twin shear stress on inner wall, psi
71. nna number of iterations required to optimize X(P)
72. nis number of iterations required to converge on
s or so
73. nib number of iterations required to converge on
tb

IV. EXPERIMENTAL SYSTEM

The experimental system can be grouped into three subsystems. These are:

- a) nozzle
- b) air supply system
- c) liquid injection system

Each subsystem is described in the following sections.

Figure 11 is an overall system schematic.

A. NOZZLE

The nozzle has a convergent-divergent flow passage. It is 12 inches long with a variable exit area. The exit area can be varied from .45313 square inches to .84375 square inches. The pivot point is located 1 inch above the throat. This causes the throat to vary when the exit is varied. Since the change in the throat is negligible, it will be considered to be constant. It has a throat area of .45 square inches. The inlet area is 1.625 square inches. The throat is located 4 inches from the inlet. The nozzle is constructed by sandwiching two 1/2 inch thick machined aluminum nozzle profile plates between 1/2 inch plexiglas plates (Fig. 12). The aluminum nozzle plates are located at the end of a 30 inch long entry section and are easily adjustable. Figure 13 shows a close up of the aluminum section of the nozzle.

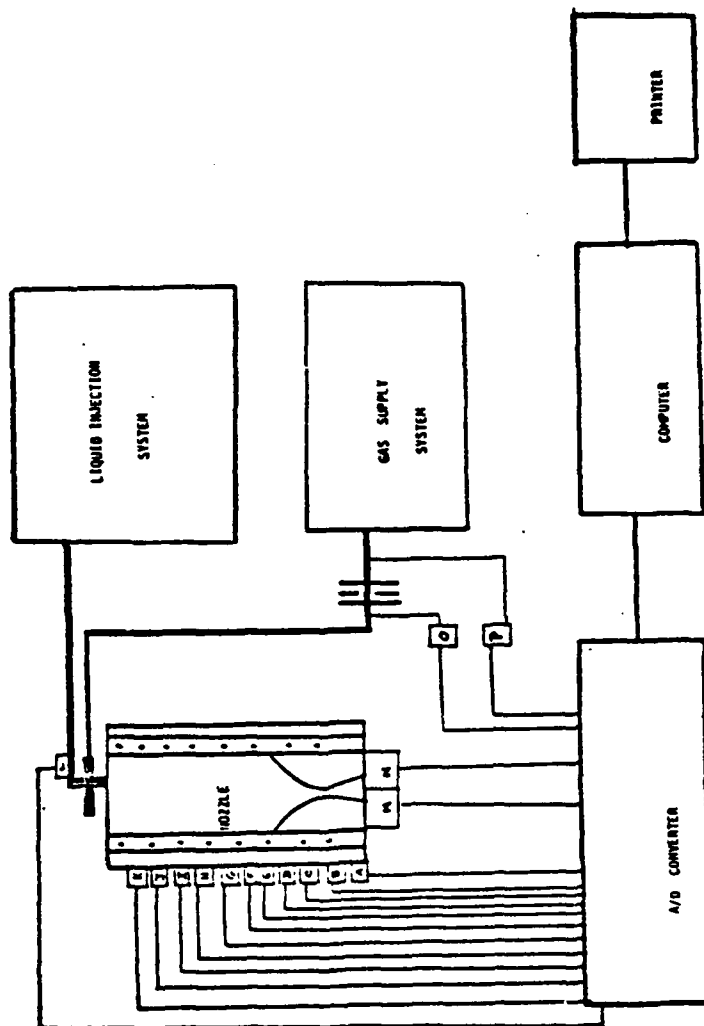


Figure 11. Experimental System Schematic

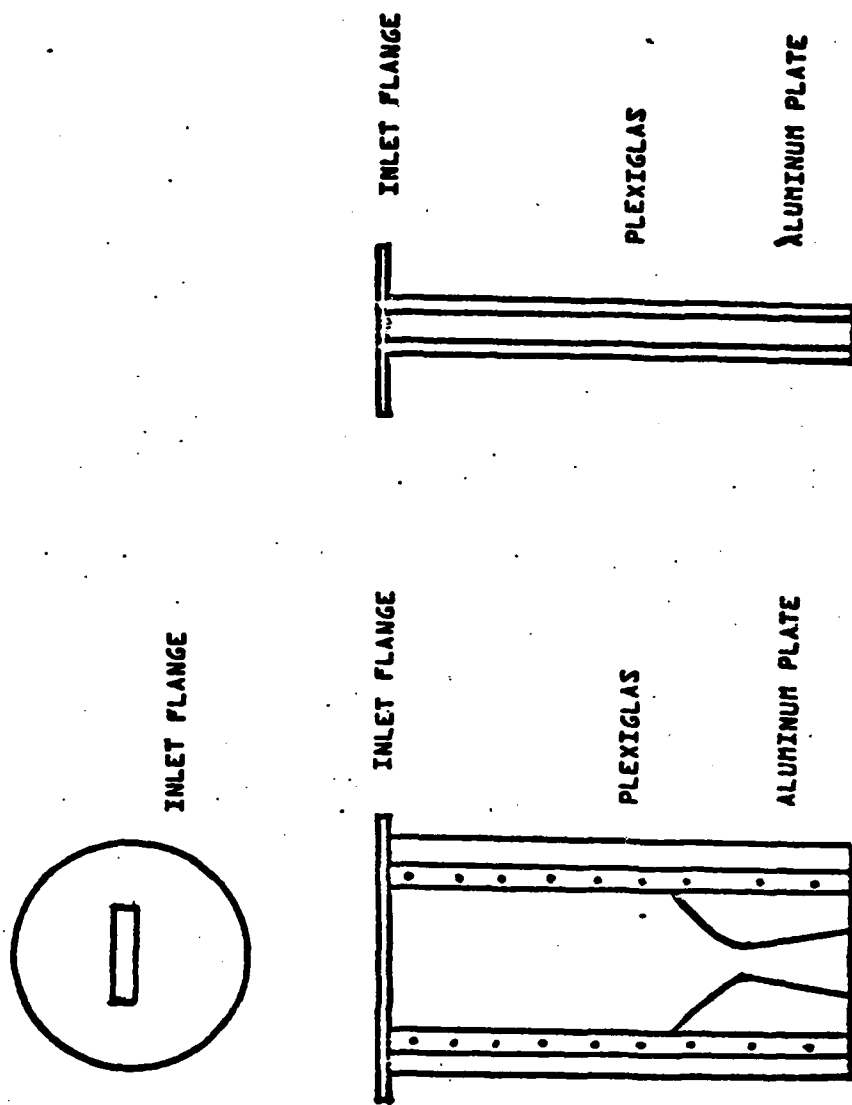
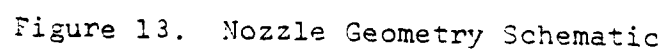


Figure 12. Nozzle Assembly Drawing



The inlet section provides the necessary space for pressure taps, liquid injection tubes, and other instrumentation and configuration options.

B. AIR SYSTEM

The air system is shown in Figure 14. Two 117 cubic feet tanks supplied with compressed air from an Ingersoll Rand two-stage air compressor provides the air storage volume required to support the nozzle's operation. Each storage tank has a pneumatically control Norgren gate valve at its exit. These valves are operated by a nitrogen actuator and controlled by a pressure regulator. The nitrogen is regulated to 40 psi control pressure which will open the Norgren valve.

The nozzle is supplied with air via a 3" i.d. pipe. The air supply to the nozzle is controlled by a solenoid actuated nitrogen operated 3 inch ball valve. The nitrogen is supplied via a regulator. By varying the nitrogen supply pressure to the ball valve, supply air pressure to the nozzle can be controlled. Air flow to the nozzle is measured with a standard ASME orifice plate. Figure 15 shows the dimensions of the orifice. The orifice is a model D-10512 with a 0.920 inch bore.

C. LIQUID INJECTION SYSTEM

The liquid injection system, Figure 16, is supplied by house water and is further pressurized by an Aurora electric

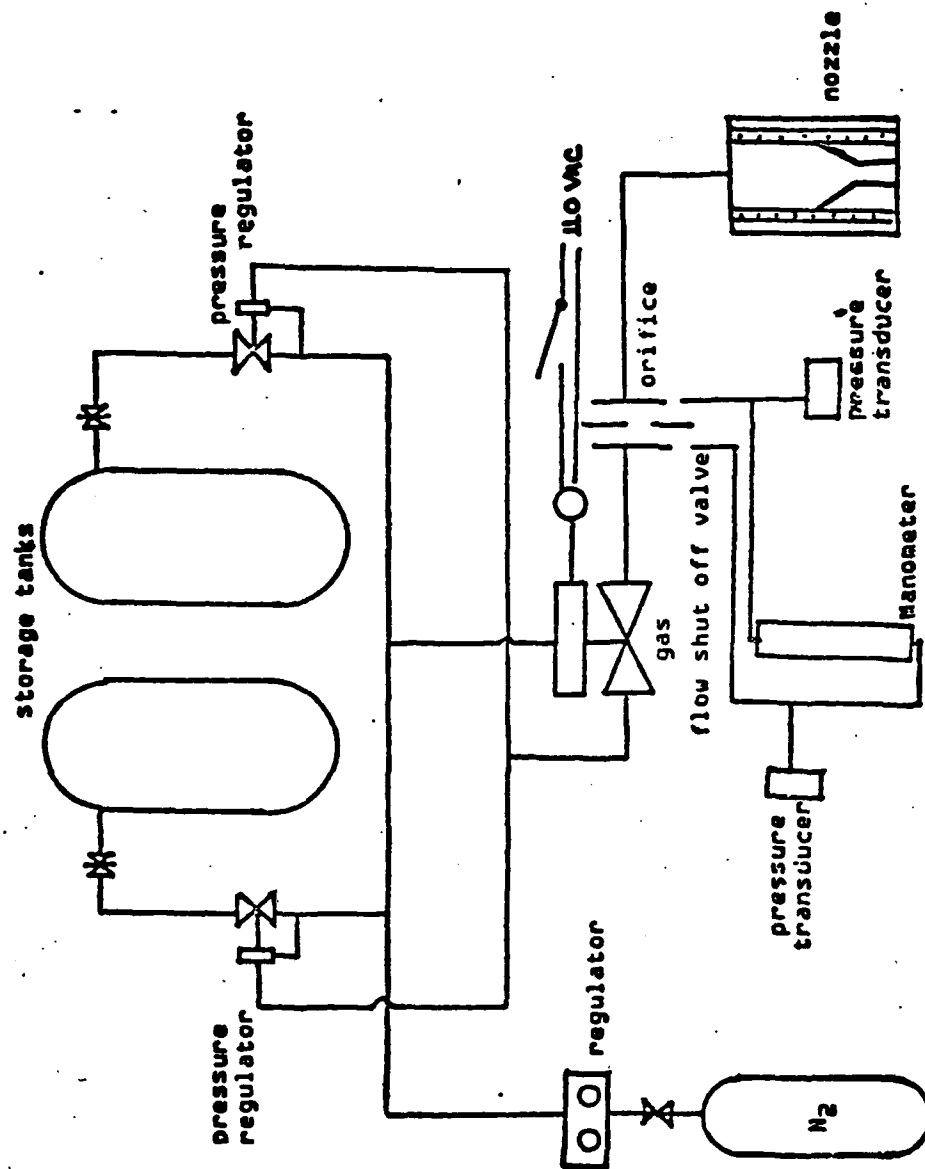


Figure 14. Orifice Schematic

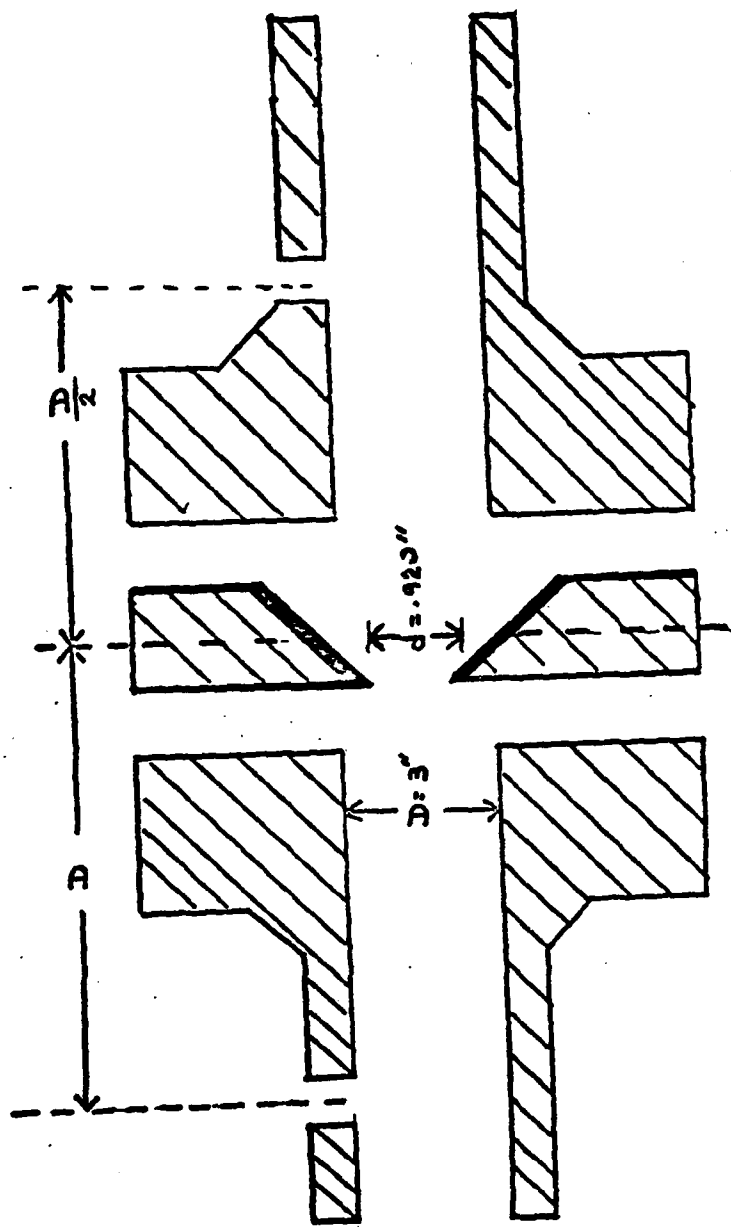


Figure 15. Air System Schematic

pump model/type 317696. The pump is rated at 50 gallons per min. The pump supplies pressurized water to the injection tube via two flowmeters (rotometers). The flowmeters are F&P co. precision bore flowrator tubes. One rotometer is rated at 1 to 12 gallons per min. and the other at .6 gallons per min.

The liquid injector is a 0.25 inch brass tube inserted in the 3" i.d. air supply pipe just upstream of the flange connection to the test section. The injector tube is drilled with sixteen 1/16th inch diameter holes facing the test section entrance. The drilled holes were made as small as possible consistent with achieving a significant liquid mass flow rate.

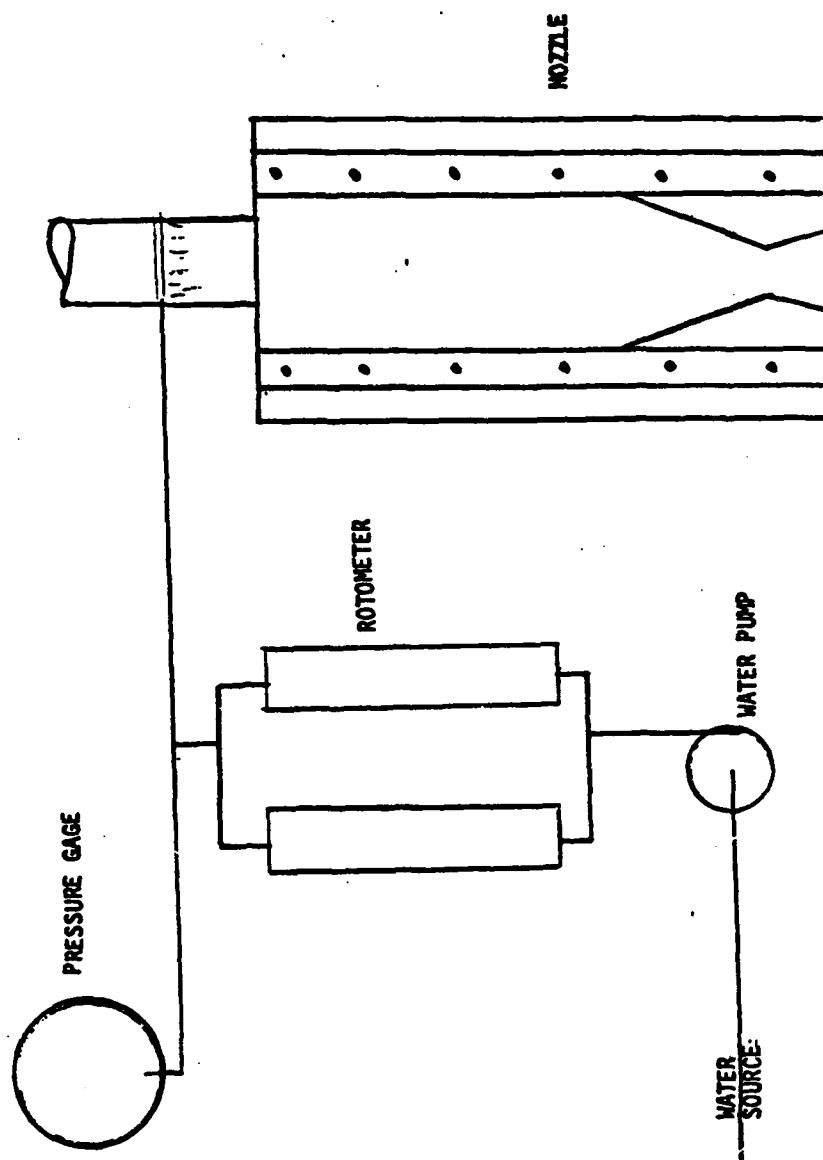


Figure 16. Liquid Injection System Schematic

V. INSTRUMENTATION SYSTEM

The instrumentation system is designed for automated data acquisition, analysis, and display. A schematic is depicted in Figure 17. Where ever possible, test operations and sequencing are under direct computer control. Each parameter measured involves an appropriate transducer, excitation source, and calibration procedure. The major instrumentation sub-systems are:

- (A) Pressure Measuring Transducers
- (B) Nozzle Thrust Force Block
- (C) Flow Measurement Devices
- (D) Data Acquisition/Control System

A. PRESSURE MEASURING TRANSDUCERS

Pressure measurements are made in fourteen locations throughout the experimental apparatus. Eleven Micro Switch 140PC pressure transducers model PK 87633 are placed on the nozzle assembly to measure pressure at various axial positions. Specifications for this model transducer are shown in Figure 18. The first pressure tap is located at one half inch from the inlet along the axis of the nozzle. The remainder are placed at one inch intervals toward the nozzle exit. These pressure taps are connected to the pressure transducers via a 1/4" o.d. plastic tubing. The

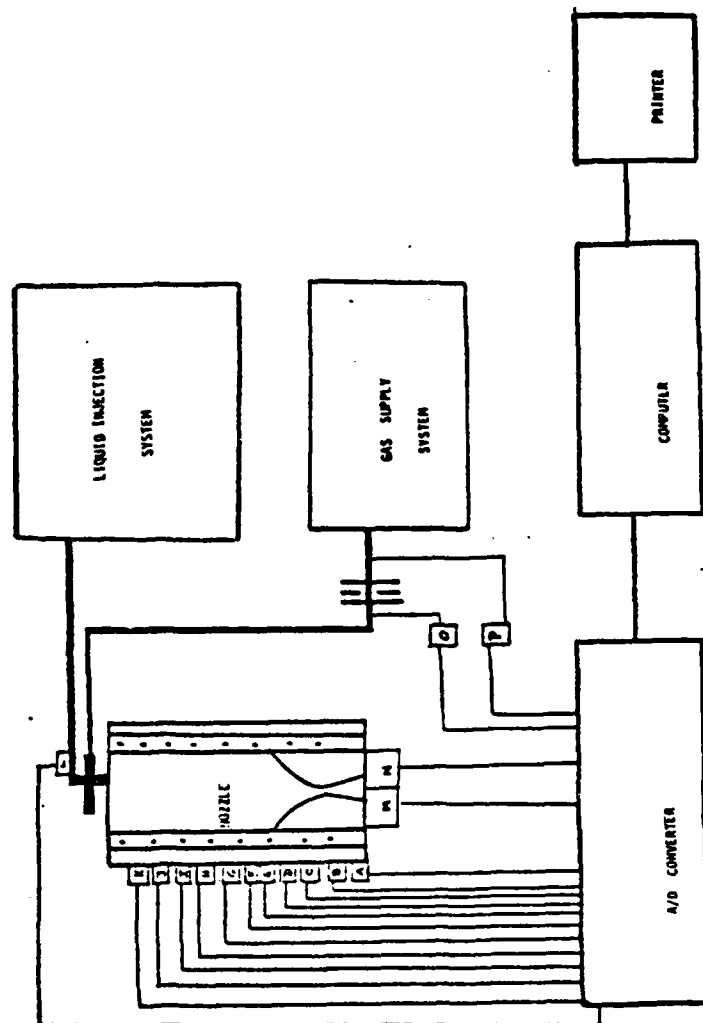
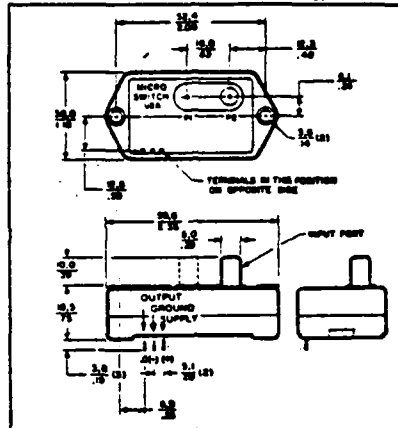


Figure 17. Instrumentation System Schematic

MOUNTING DIMENSIONS (for reference only)



140PC SPECIFICATIONS at 0.5 0.01VDC, 25°C

ALL LISTINGS

PARAMETER	PRESSURE RANGE (psi)	Min.	Typ.	Max.	UNITS
F.S.O. (Full Scale Output)*	All	4.85	5.00	5.15	Volts
Null Offset	All	0.95	1.00	1.05	Volts
Excitation	All	8.00		20.0	VDC
Output Current Source	All	10.0			mA
Supply Current (10K ohm load)	All	5.0	6.0		mA
Overpressure	0-1 0-5 0-15 0-15/0-30(L)			20 25 45 60	psi
Operating Temperature		-55°C to +125°C (-65°F to +257°F)			

ELECTRICAL AND PRESSURE CONNECTIONS

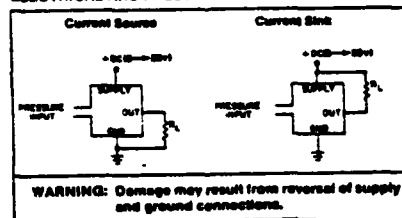


Figure 18. 140PC Pressure Transducer Specifications

pressure transducers are mounted on the side panel of the nozzle. See Figure 19 for locations. Each transducer has an identification letter corresponding to its position. Table I is the ID code for the transducers.

At the nozzle inlet a Micro Switch 200PC pressure transducer model PK 87690 is placed in the inlet line to measure the dual-phase mixture inlet pressure. Specifications for this model transducers are shown in Figure 19. Gas supply pressure is measured using a 200PC pressure transducer located at the orifice outlet. This transducer is also used to help measure differential pressure across the orifice. Another 200PC transducer is placed at the inlet of the orifice. The data acquisition program calculates the orifice differential pressure and then the air mass rate using the output from the above two transducers.

The pressure transducers provide output voltage proportional to applied pressure. These operate from a single, positive supply voltage ranging from 8 to 20 VDC. The supply voltage was maintained at 15VDC.

Each pressure transducer sends back a signal proportional to the input pressure. The signal is converted in the A/D converter to a digital signal which can be read by the HP9826. The pressure transducers were calibrated using a known pressure source. Appendix F depicts a sample calibration program written for a HP9826 computer. This program reads ten



* If S.D. is the square difference between end points (full and full pressure outputs).

ELECTRICAL AND PRESSURE CONNECTIONS



Table I. ID Code for Instrumentation

<u>Letter Designation</u>	<u>Description</u>
A	10.5" from inlet
B	9.5" from inlet
C	8.5" from inlet
D	7.5" from inlet
E	6.5" from inlet
F	5.5" from inlet
G	4.5" from inlet
H	3.5" from inlet
I	2.5" from inlet
J	1.5" from inlet
K	0.5" from inlet
L	inlet nozzle pressure
M	force-block signal
N	force-block excitation voltage
O	orifice exit pressure
P	orifice inlet pressure

$$\text{Pressure} = A + B(\text{volts}) + C(\text{volts})^2 + D(\text{volts})^3$$

pressure transducer	A	B	C	D
A	-6.049	3.926	-.17336	.01121
B	-7.059	4.234	-.22921	.01415
C	-.6491	3.934	-.16025	.00945
D	-7.056	4.692	-.34118	.01898
E	-7.009	4.152	-.20717	.01237
F	-6.389	3.933	-.16055	.00951
G	-6.589	3.956	-.16427	.00961
H	-6.588	3.908	-.15806	.00942
I	-6.595	3.949	-.16270	.00957
J	-6.353	3.927	-.16509	.00998
K	-6.203	3.846	-.14697	.00880
L	-20.207	11.157	-.08591	.00492
O	-20.668	11.068	-.07562	.01118
P	-18.827	11.928	-.17715	.01101

Figure 20. List of Polynomial Coefficients
for Pressure Transducer

values of pressure for each 140PC transducer and gives the mean and standard deviation for each from 0 to 45 psi in 5 psi increments. Appendix G depicts sample output of the program for a given pressure. Appendix H illustrates the program used to calibrate the 200PC pressure transducers. The program works as above, except values are taken from 0 to 60 psi at 5 psi increments.

Data obtained during calibration source pressure is plotted for each pressure transducer (Appendix I). Each plot was curve fitted with a third order polynomial. Figure 20 shows coefficients of the polynomial. These polynomials are used in the data acquisition/control program to convert transducer readings to pressure readings.

B. NOZZLE THRUST FORCE-BLOCK

The thrust produced by the nozzle was employed to deduce the nozzle exit velocities. The thrust was measured by instrumenting a target plate in the exit flow field. Appropriate screens were installed to prevent liquid "bounce back." The jet momentum force on the target is acquired by a balance beam system shown in Figure 21. A Kistler-Morse force block provides an analog signal proportional to the nozzle jet momentum.

The calibration of the force-block was completed by placing known weights on the force-block side of the balance

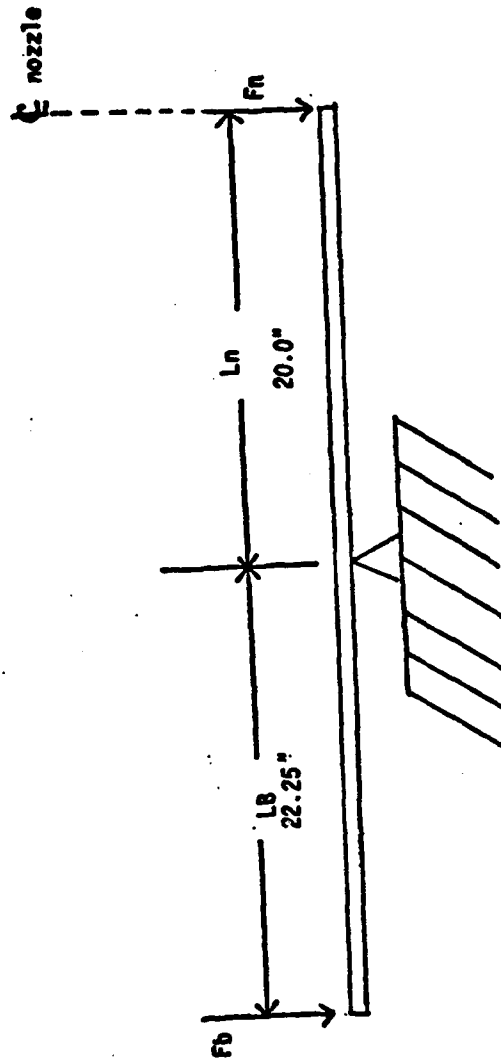


Figure 21. Force-Block Balance Beam Diagram

beam, (Figure 21), and recording the voltage produced.

Appendix J shows results of this calibration. These results were curve fitted with a third order polynomial. The following is the volt-to-force conversion polynomial:

$$FB = (1/453.6)*(-234.396+48715.28*VOLTS-11261.54*VOLTS+33899.3*VOLTS)$$

$$FN = FB*LB/LN$$

$$FN = \text{FORCE AT NOZZLE EXIT (LBF)}$$

$$FB = \text{FORCE AT FORCE BLOCK (LBS)}$$

$$LB = \text{LENGTH FROM PIVOT TO FORCE BLOCK(IN.)}$$

$$LN = \text{LENGTH FROM PIVOT TO DIRECTIONAL OBSTRUCTION(IN.)}$$

C. FLOW MEASUREMENT

1. Air mass flow measurement for the system achieved by measuring the inlet and outlet pressure on a D-10512 orifice with a 0.92 inch bore. For details on orifice see Section III. Air flow is calculated in the data acquisition program. The program uses the following equation obtained from References 4 and 5.

$$M_{\text{air}} = K A_2 Y \sqrt{2G_c \rho_1 (P_1 - P_2)} \quad \text{where}$$

$$K = CE$$

$$E = 1/(1-B^4)^{1/2}$$

$$B = d/D$$

$$C = 0.60$$

$$A_2 = \text{AREA OF ORIFICE} = \frac{\pi D^2}{4} \quad \text{FT}^2$$

$G_c = 32.2 \text{ FT/SEC}^2$

$\rho_1 = \text{DENSITY LBM/FT}^3$

$P_1 = \text{PRESSURE AT INLET IN LBF/FT}^2$

$P_2 = \text{PRESSURE AT LUTLET IN LBF/FT}^2$

$M_{\text{air}} = \text{AIR MASS FLOW RATE LBM/SEC}$

The discharge coefficient C is the factor that accounts for losses through the orifice. Since the values of C varies from .62 to .60 for R_d number from 10^4 to 10^7 with $\beta = .3$, C will be considered constant.

2. Water flow measurement was made using two rotometers. Calibration of the rotometers were made by measuring the time for a given weight of fluid flow. The mass flow rate was calculated and plotted versus the rotometer reading. Appendix K is the plot of the results. The plot was curve fitted with a third order polynomial. The following is that polynomial:

$$MH20 = -.0063268 + .002097278 * RR - .00000658 * RR^2 + 1.1X10 * RR^3$$

where $RR = \text{rotometer reading}$

D. DATA ACQUISITION/CONTROL SYSTEM

The heart of the data acquisition system is a HP9826 small computer. The HP9826 communicates via a Hewlett Packard 3497A data acquisition/control system. This system gathers data from the pressure transducers and nozzle thrust force-block. It converts the analog signal to digital data, and stores the data in memory. Figure 22 shows a pressure transducer to A/D converter channel connection, and Figure 23

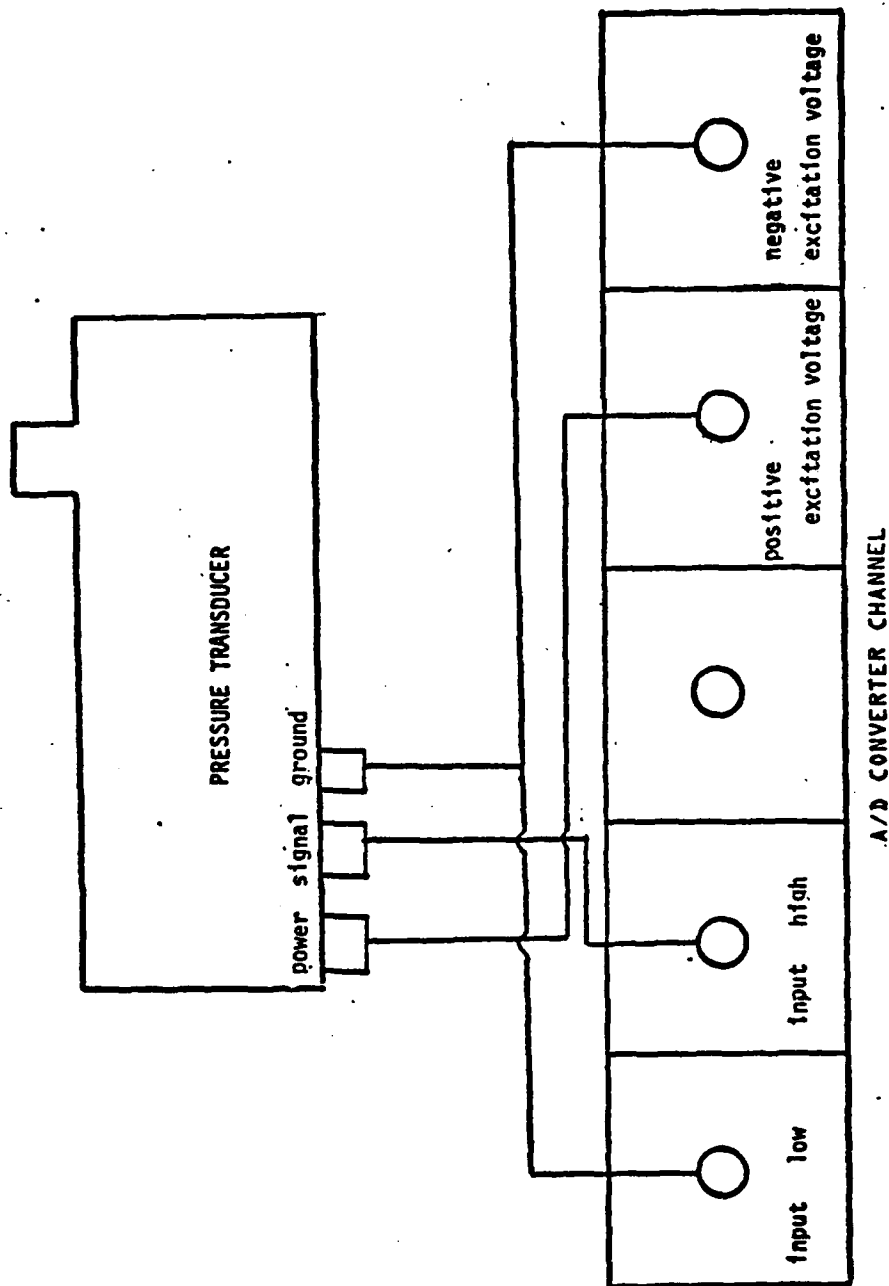


Figure 22. Transducer-to-Channel Connection

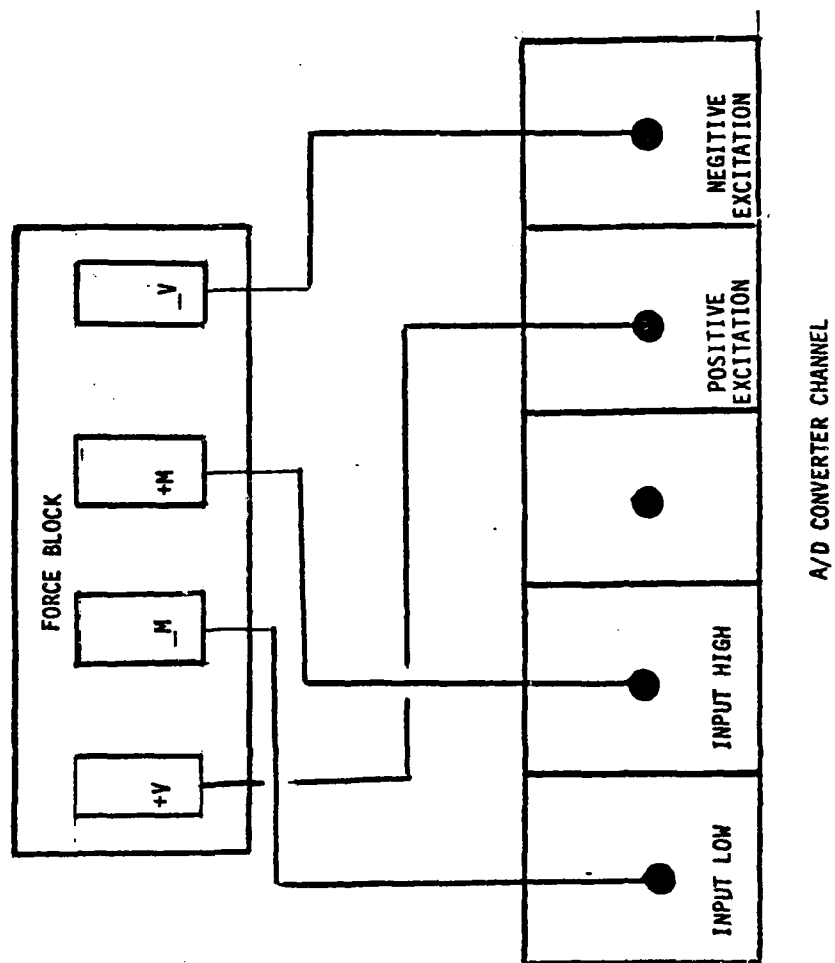


Figure 23. Force-Block-to-Channel Connection

CHANNEL 7	CHANNEL 8	CHANNEL 9	CHANNEL 16	CHANNEL 17	CHANNEL 18
6	H	I		EXCITATION VOLTAGE	
CHANNEL 4	CHANNEL 5	CHANNEL 6	CHANNEL 13	CHANNEL 14	CHANNEL 15
D	E	F	O	P	M,N
CHANNEL 1	CHANNEL 2	CHANNEL 3	CHANNEL 10	CHANNEL 11	CHANNEL 12
A	B	C	J	K	L

BUS CONNECTION TO A/D CONVERTER

Figure 24. Bus Connection-to-A/D Converter Connections

the force-block to A/D converter channel connection. Figure 24 depicts the current connection to the data acquisition/control system bus from the sensors.

The data acquisition and analysis program is a fully automated system which integrates the use of the HP9826, HP3497A, and all sensor devices. The program can be divided into three sections:

1. Data Acquisition - Data is acquired from 15 different sensor devices. All system parameters are initialized and configuration types are inputted into the computer memory for future use. Ten readings are made on each sensor and a mean value for each is calculated and stored in memory.
2. Data Conversion - Input data conversion is completed by using calibration information obtained on each pressure transducer and force-block. The program takes the stored mean values from memory, and converts the input voltage to a pressure or force using the third order calibration polynomial for each sensor.
3. Data Analysis - This is the heart of the program in that all performance parameters are calculated and printed. It uses all of the above information to calculate mass flow rate of water, mass flow rate of air, total mass flow rate, thrust, mixture

ratio, exit velocity and all pressure information. The following equations are also used:

$$\text{Exit Velocity} = (\text{Thrust} / \text{Total Mass Flow Rate}) * g_c (\text{ft/sec})$$

$$\text{Mixture Ratio} = \dot{m}_{\text{water}} / \dot{m}_{\text{air}}$$

$$\text{Total Mass Flow Rate} = \dot{m}_{\text{water}} + \dot{m}_{\text{air}} \left(\frac{\text{lbm}}{\text{sec}} \right)$$

Appendix L is the data acquisition program and sample output.

VI. EXPERIMENTAL RESULTS

Six test runs were made using the equipment described in the previous sections. It was found that the inlet pressure to the nozzle remained nearly constant for each setting. The testing was conducted by maintaining a constant air control valve setting and varying the mass flow rate of water. The regulator outlet pressure of the nitrogen bottle which controls the air valve was set at 30 psi and 15 psi. These settings correspond to nozzle inlet pressure of 35 ± 1.0 psi and 29 ± 1.0 psi respectively. For each nozzle inlet pressure, experimental data was obtained at varying mass ratios of water-to-air in the range of 2 to 13. These experiments were conducted with three discharge area ratios (A_{inlet}/A_{out}). Those ratios are: 3.586, 2.600, and 1.9259. The results of the six experimental tests are displayed in Tables II through VII.

Inlet air was limited to a maximum value of 40 psi. The 140PC pressure transducer's maximum output pressure is 40 psi, (see information on 140PC transducer Figure 21). The experimental system has the ability to reach higher pressures. The present configuration may be operated to 70 psig.

Table II. Experimental Data Exit Area = .45313 sq. in.
Inlet Pressure = 29+1.5psi

NOZZLE GEOMETRY 0.45313 SQ.IN*****PRESSURE 29.00000 PSI*****									

*****MASS FLOW RATE*****									
*****TOTAL AIR*****									
*****WATER*****									
*****0.04220*****									
*****0.01830*****									

*****MASS RATIO*****									
*****2.30000*****									
*****1998.00000*****									
*****1950.39900*****									
*****VELOCITY*****									
*****1300.00000*****									
*****1243.30000*****									
*****1001.89900*****									
*****830.00000*****									
*****760.30000*****									
*****695.50000*****									
*****643.19990*****									
*****602.50000*****									
*****620.89990*****									
*****550.00000*****									
*****4.37000*****									
*****4.41000*****									
*****4.29000*****									
*****4.24000*****									
*****4.16000*****									
*****4.11000*****									
*****3.92000*****									
*****3.75000*****									
*****THRUST*****									

Table III. Experimental Data Exit Area = .45313 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY		0.45313 SQ. IN.		PRESSURE		35.00000 PSI			
				</					

Table IV. Experimental Data Exit Area = .62500 sq. in.
Inlet Pressure = 29 $\pm$ 1.5psi

MUZZLE GEOMETRY 0.62500 SQ. IN*****PRESSURE 29.00000 PSI*****									
*****MASS FLOW RATE*****		*****WATER RATIO*****		*****EXPERIMENTAL REAL EXIT VELOCITY*****		*****THRUST*****			
TOTAL	AIR			VELOCITY	VELOCITY				
0.06040	0.01820	0.04220	2.32000	1834.50000	1880.10000		3.44000		
0.10150	0.01810	0.08346	4.61000	1159.00000	1201.19900		3.66000		
0.13660	0.01800	0.11860	6.59000	937.80000	953.60000		3.98000		
0.16650	0.01790	0.14860	8.32000	802.19990	803.10000		4.15000		
0.19220	0.01770	0.17450	9.85000	688.60000	749.19990		4.21000		
0.21510	0.01780	0.19730	11.10000	643.19990	702.10000		4.30000		
0.23550	0.01740	0.21810	12.60000	589.89990	670.10000		4.32000		

Table V. Experimental Data Exit Area = .62500 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY 0.62500 SQ.IN*****PRESSURE 35.00000 PSI									

*****MASS FLOW RATE*****									
*****TOTAL AIR*****									
*****WATER*****									
*****0.02900 0.04220*****									
*****MASS RATIO*****									
*****2.02000 2165.89900 2210.30000*****									
*****EXPERIMENTAL REAL EXIT VELOCITY*****									
*****THRUST*****									
*****4.24000*****									
0.13910	0.02050	0.11860	5.77000	1097.10000	1153.10000	4.74000			
0.16900	0.02050	0.14860	7.27000	940.50000	998.39990	4.94000			
0.19980	0.02030	0.17450	8.60000	832.89990	885.30000	5.04000			
0.21760	0.02030	0.19730	9.72000	765.34990	801.60000	5.17000			
0.23820	0.02010	0.21810	10.87000	720.50000	753.39990	5.32000			
0.25770	0.01890	0.23790	12.03000	674.80000	700.30000	5.40000			

Table VI. Experimental Data Exit Area = .84375 sq. in.
Inlet Pressure = 29±1.5psi

[illegible]

Table VII. Experimental Data Exit Area = .84375 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY		0.84375 SQ. IN.		PRESSURE		35.00000 PSI	
TOTAL		MASS		EXPERIMENTAL		REAL	
0.06350		0.02130		0.04220		1.98000	
0.10440		0.02090		0.08346		3.99000	
0.13900		0.02040		0.11860		5.82000	
0.16890		0.02010		0.14860		7.31000	
0.19460		0.02010		0.17450		8.70000	
0.21690		0.01960		0.19730		10.10000	
0.23810		0.02000		0.21810		10.90000	
0.25780		0.01980		0.23790		12.00000	

VII. DISCUSSION

The results of this investigation are in two parts; the experimental test results and the computer outputs based on initial conditions similar to those of the experiments.

The key variables are: the nozzle overall area ratio A_R , liquid/air mass flow ratio, nozzle exit velocity, nozzle supply pressure, nozzle thrust, and the nozzle axial pressure profiles. Figures 25 through 34 present the exit velocity vs. mass ratio results for the experiment and computer output. Figure 29-34 illustrates the comparison of the experimental results and the corresponding computer output. Figure 35 illustrates the variation of nozzle thrust with mass ratio for different area ratios and inlet pressures. Figures 36 and 37 present the axial pressure profiles.

In all cases the exit plane velocity decreases as the liquid/air mass ratio is increased. In the low mass ratio range (i.e., less than ≈ 5) the velocity decrease is very pronounced. Past a mass ratio value of ≈ 10 the velocity of the mixture becomes relatively insensitive to the mixture ratio. This is as would be expected. For the same inlet conditions there is a fixed amount of energy available for conversion to its kinetic form. In the nozzle process the energy is conserved and thus an increasing mass ratio is

manifested in a decreasing exit plane velocity. In all cases the agreement between the experimental tests and the analytic predictions are within $\approx 10\%$.

There also was apparent a consistent trend with respect to the nozzle area ratio and the exit velocity. As the nozzle exit area was increased the exit velocity of the mixture decreased. The effect seems to be more pronounced at the lower mass ratios. This trend was confirmed by observations of the actual flow field in the nozzle. If the exit area was increased significantly (beyond the max area used in those tests) there was evident an abrupt and severe separation within the diverging portion of the nozzle passage. This was accompanied by a drastic decrease in exit velocity as evidenced by the output from the thrust target. It appears that the diverging portion of the nozzle, at a certain point, starts to behave as a subsonic diffuser and hence experiences an adverse pressure gradient. This reasoning is in part confirmed by the pressure profiles identified in Figures 36 and 37.

The relationship between measured thrust and mass ratio (Figure 35) indicates a slightly increasing trend. This may be explained by considering the following. Each set of data points (at constant inlet air pressure and constant nozzle exit area) is developed by varying the mass ratio. This, in turn, is achieved by increasing the liquid rate by increasing the liquid supply pressure. The net result is an

increase in the liquid inlet velocity or an increase in the inlet energy level. Thus a particular data set is not truly at a constant inlet energy level but is increasing.

Perhaps the major discrepancy between the experimental test results and the computer analytic model is evidenced in the axial pressure profiles of Figures 36 and 37. It is clearly evident that at a certain point the nozzle passage reverts from a nozzle to a diffuser. This transition point occurs slightly downstream of the throat and the pressure starts to increase.

Unfortunately the computer analytic model requires a pressure profile as an input. Furthermore the pressure profile must be continuously decreasing. Thus the profile as obtained from the experiment are not directly useable. The situation is examined with the aid of Figure 38. Curve A is a typical axial pressure distribution as obtained from the experiment. Curve B is a pressure distribution used by Elliott in the application of his computer program. Pressure profiles C and D were arbitrarily defined and the exit plane velocity for each was calculated. Velocity variation in the range of 1% is evident. It appears that the final exit velocity is relatively insensitive to the actual pressure profile in the nozzle.

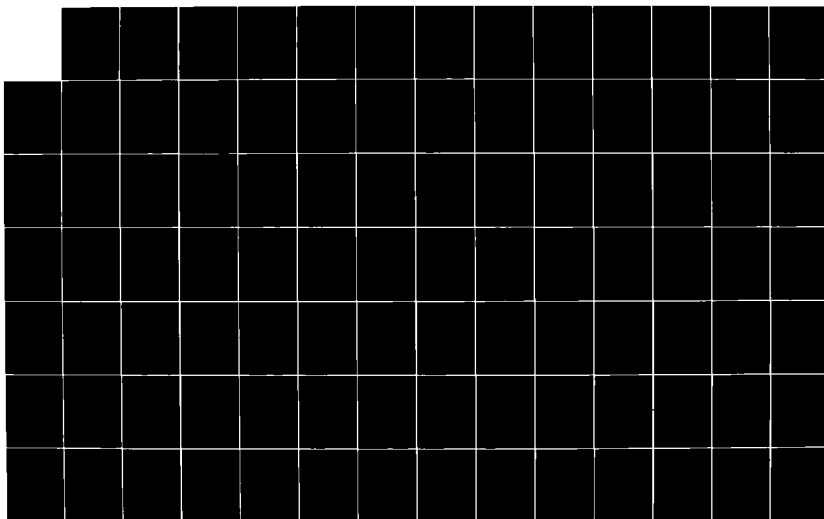
In light of the preceeding difficulty of matching the experimental pressure profile to the computer model and the

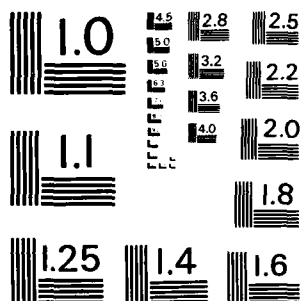
AD-A125 730 DUAL-PHASE NOZZLE FLOW(U) NAVAL POSTGRADUATE SCHOOL
MONTEREY CA T C MOLLIE OCT 82

2/3

UNCLASSIFIED

F/G 13/10 NL

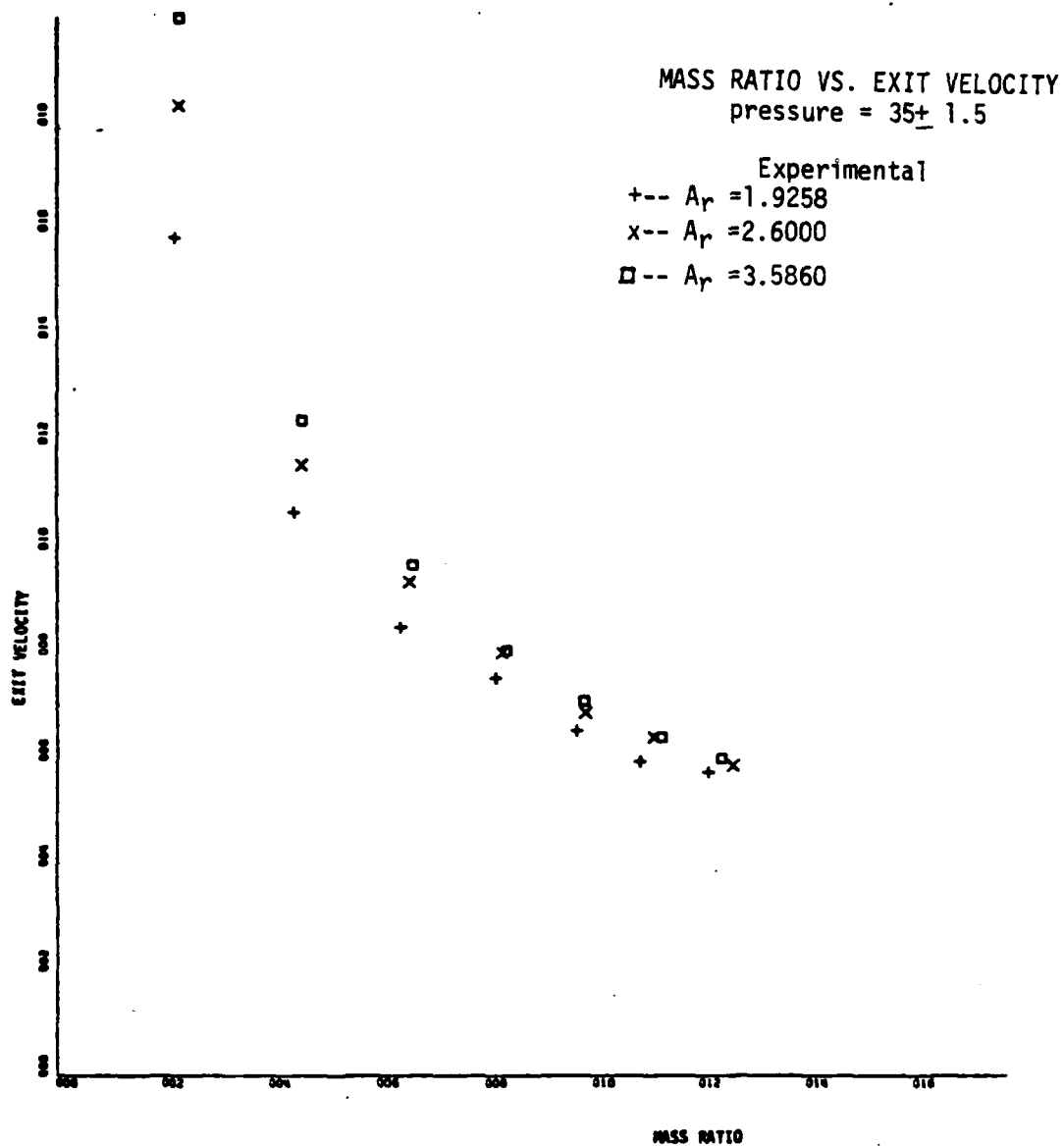




MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

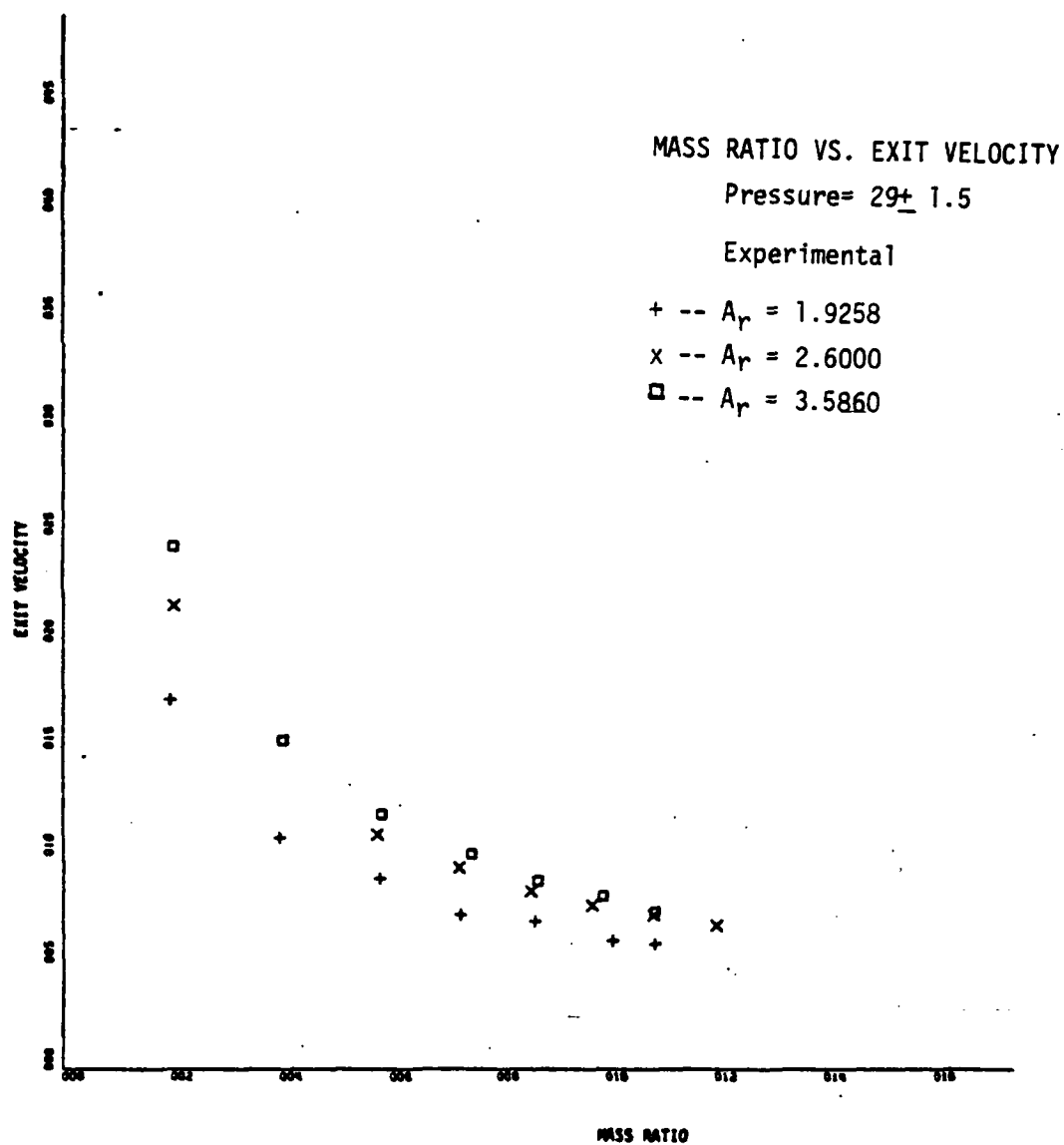
apparent flexibility of the type of pressure profile it was decided to employ a form of profile D of Figure 38. Thus instead of using the actual experimental pressure data as an input in the computer model, a profile resembling curve D was developed for each inlet pressure case.

It is apparent that within certain rather wide limits the Elliott computer model yields results that correspond within 10% to results obtained from the experiment. The general trends have been confirmed and their behavior has revealed nothing unexpected. The conclusion of Elliott [Ref. 1] has been largely substantiated. "It is very difficult to design a bad or a good dual-phase nozzle."



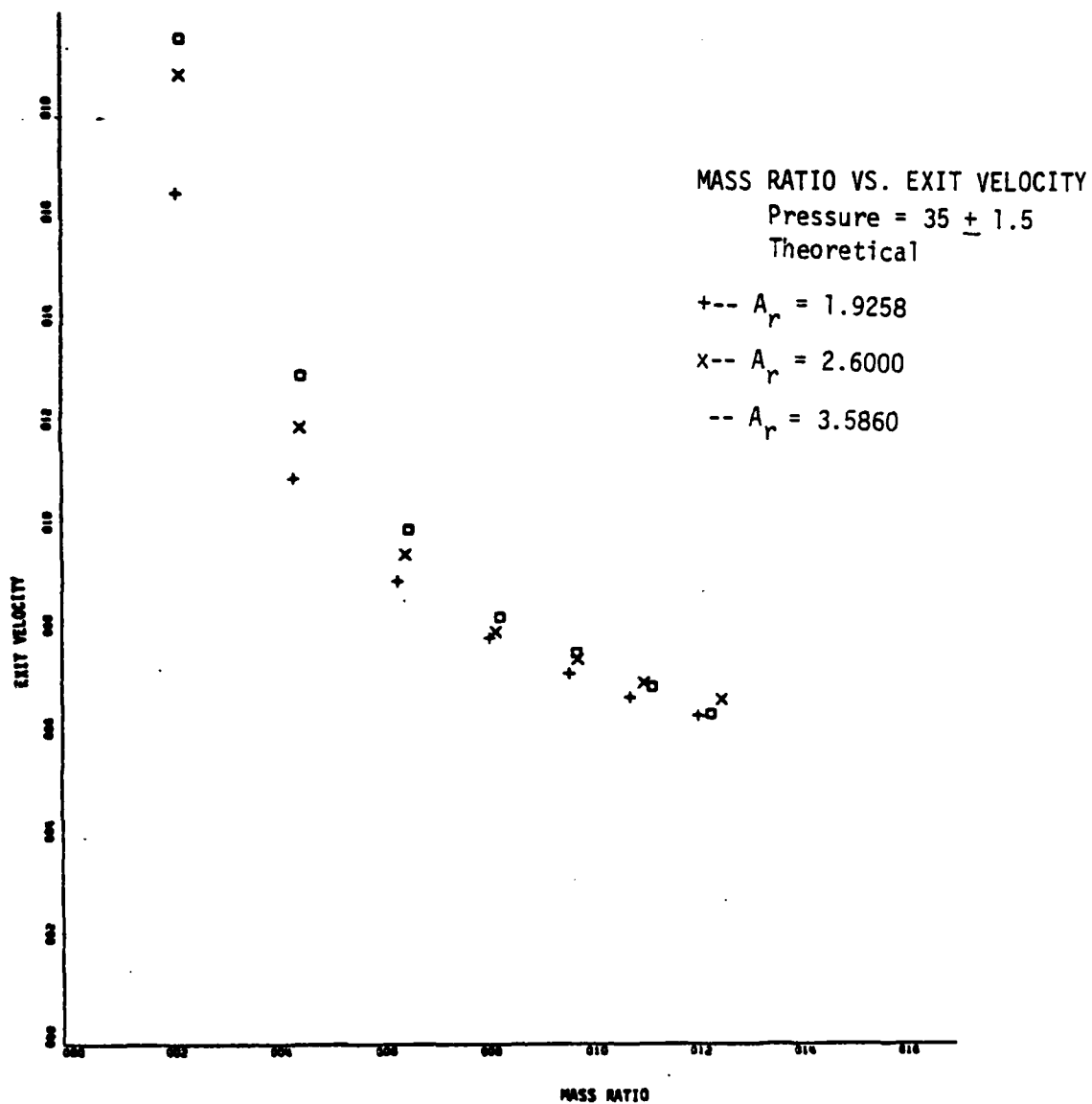
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 25. Mass Ratio vs. Exit Velocity at
Pressure = 35 ± 1.5 psi Experimental



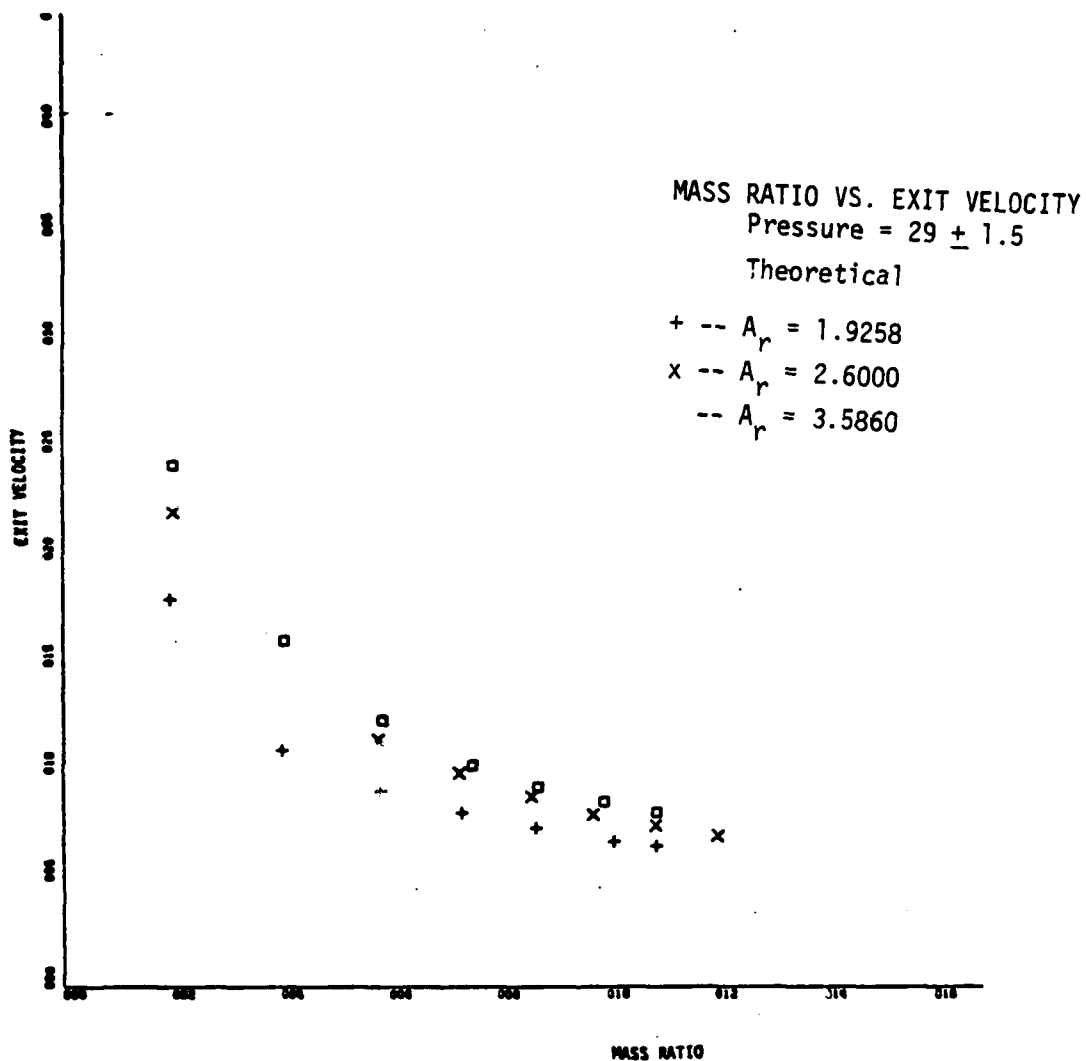
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=5.00E+02 UNITS INCH.

Figure 26. Mass Ratio vs. Exit Velocity at
Pressure = 29 ± 1.5 psi Experimental



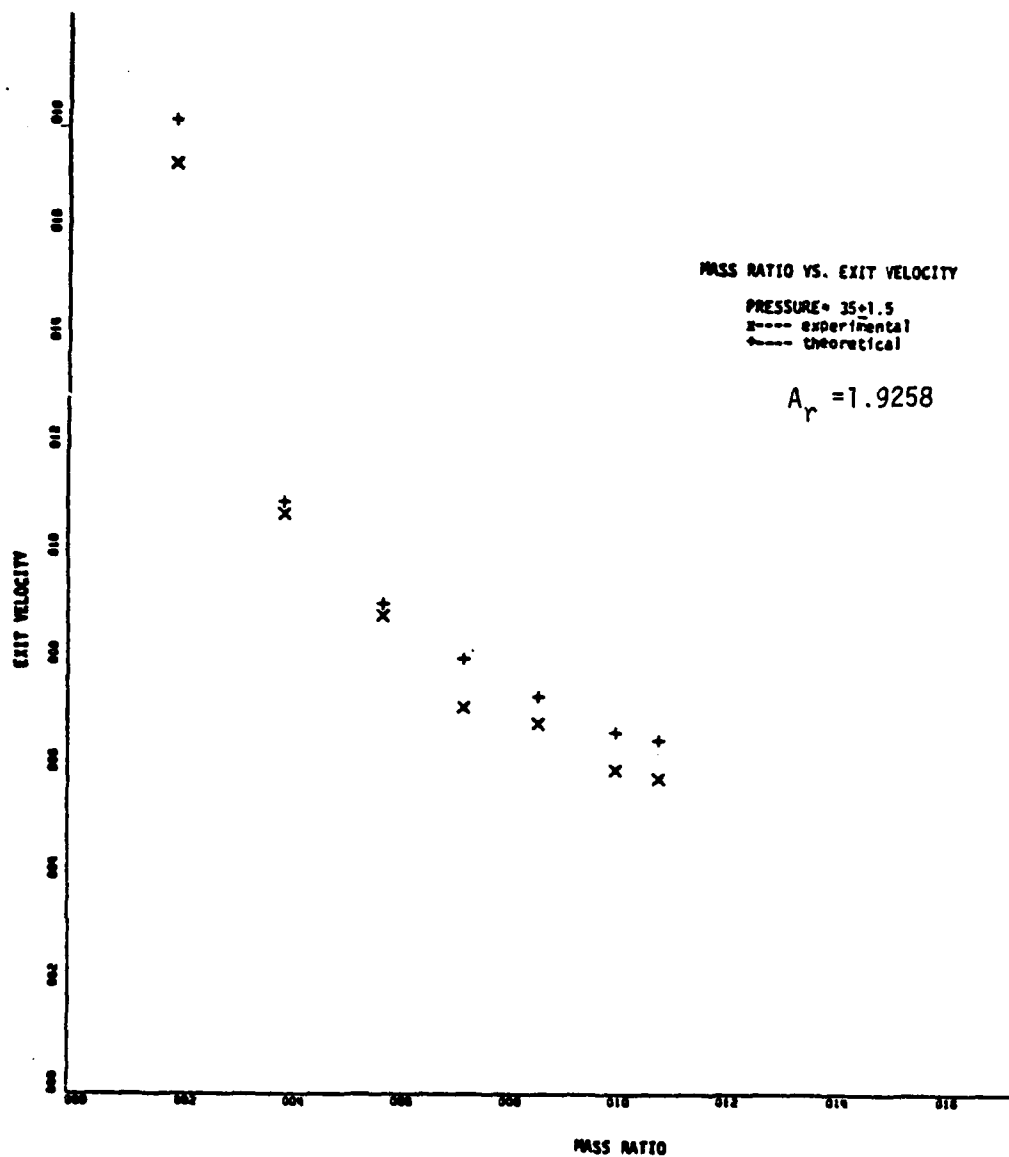
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 27. Mass Ratio vs. Exit Velocity at
Pressure = 35 ± 1.5 psi Theoretical



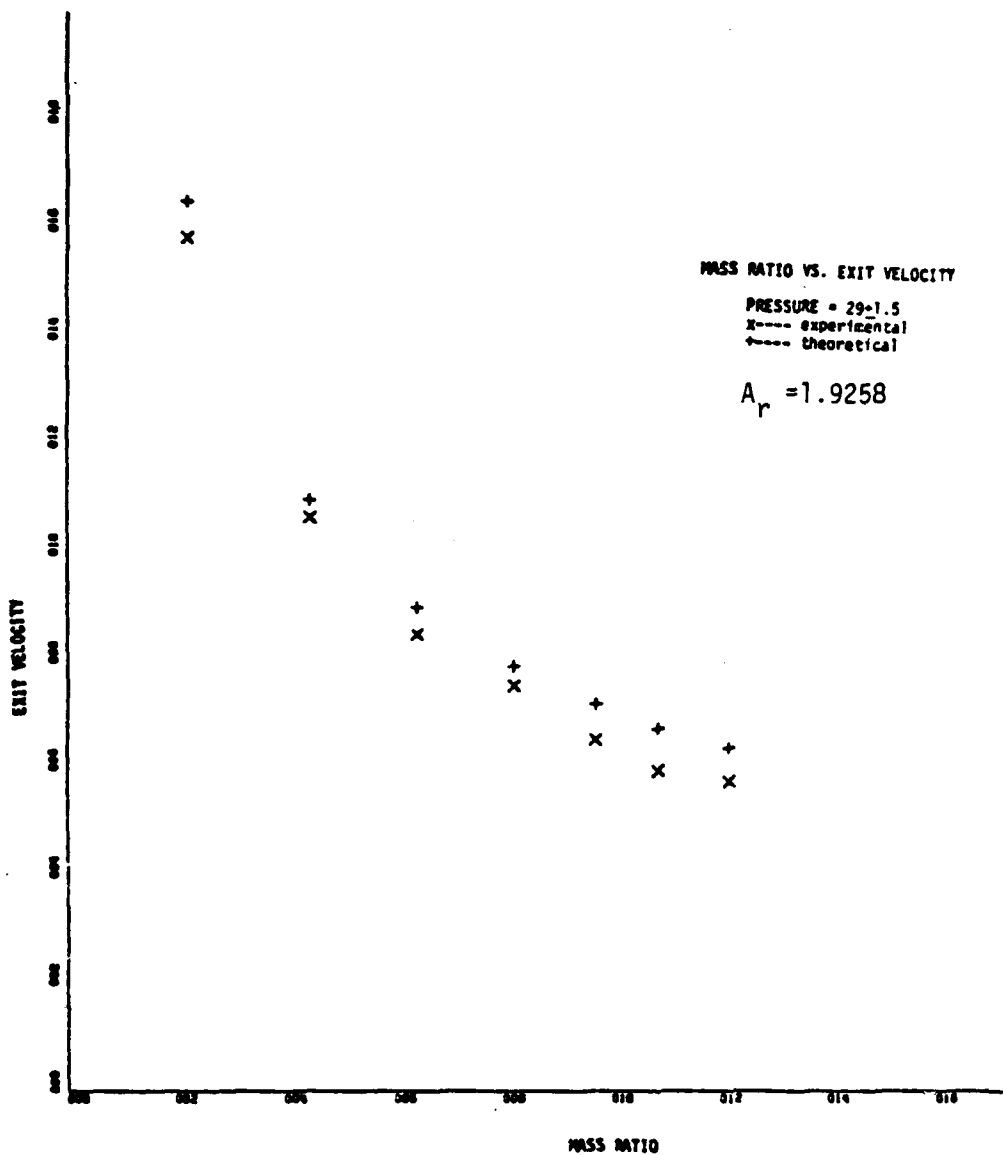
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=5.00E+02 UNITS INCH.

Figure 28. Mass Ratio vs. Exit Velocity at
Pressure = 29 ± 1.5 psi Theoretical



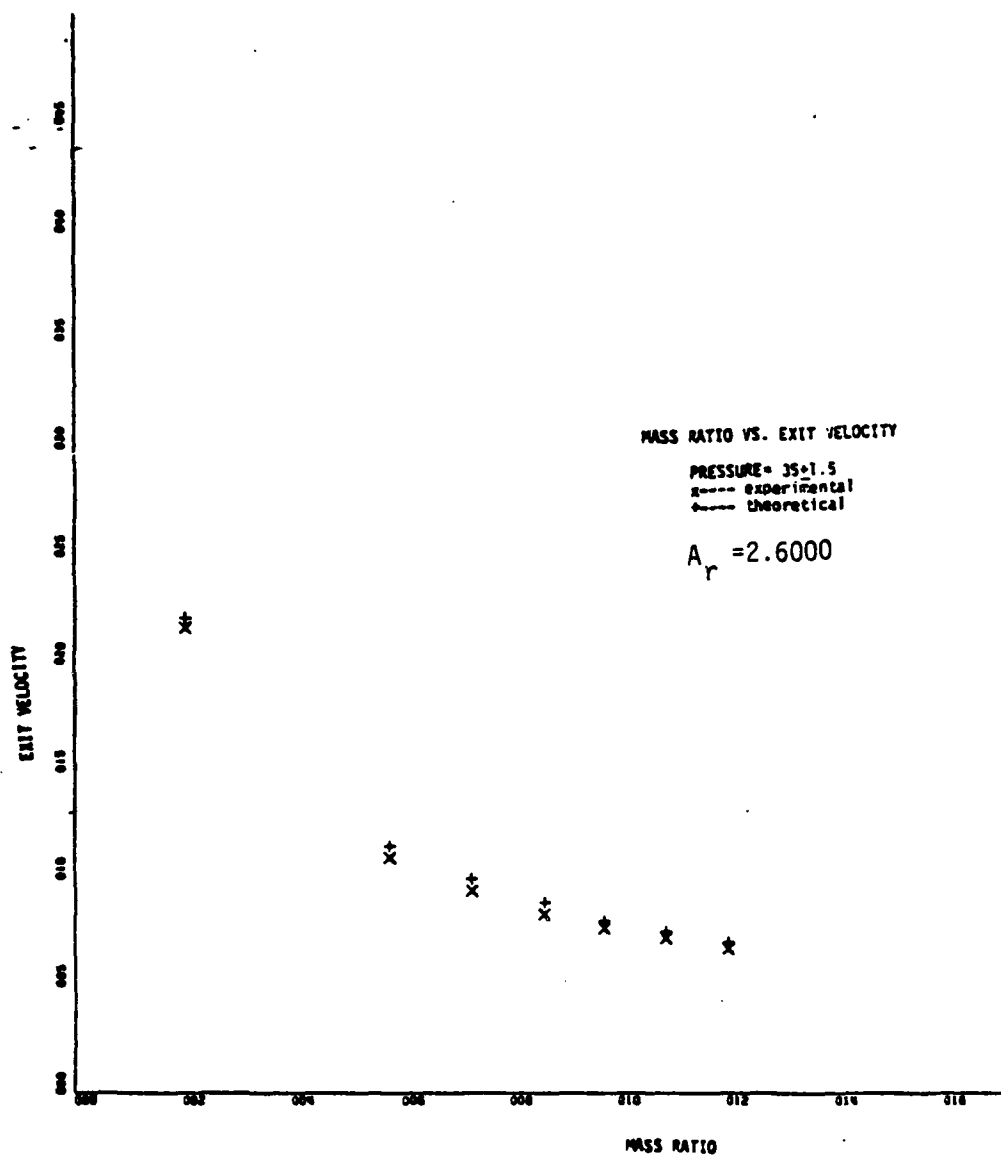
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=2.00E+02 UNITS INCH.

Figure 29. Mass Ratio vs. Exit Velocity at
 Pressure = 35±1.5psi Area = .34375



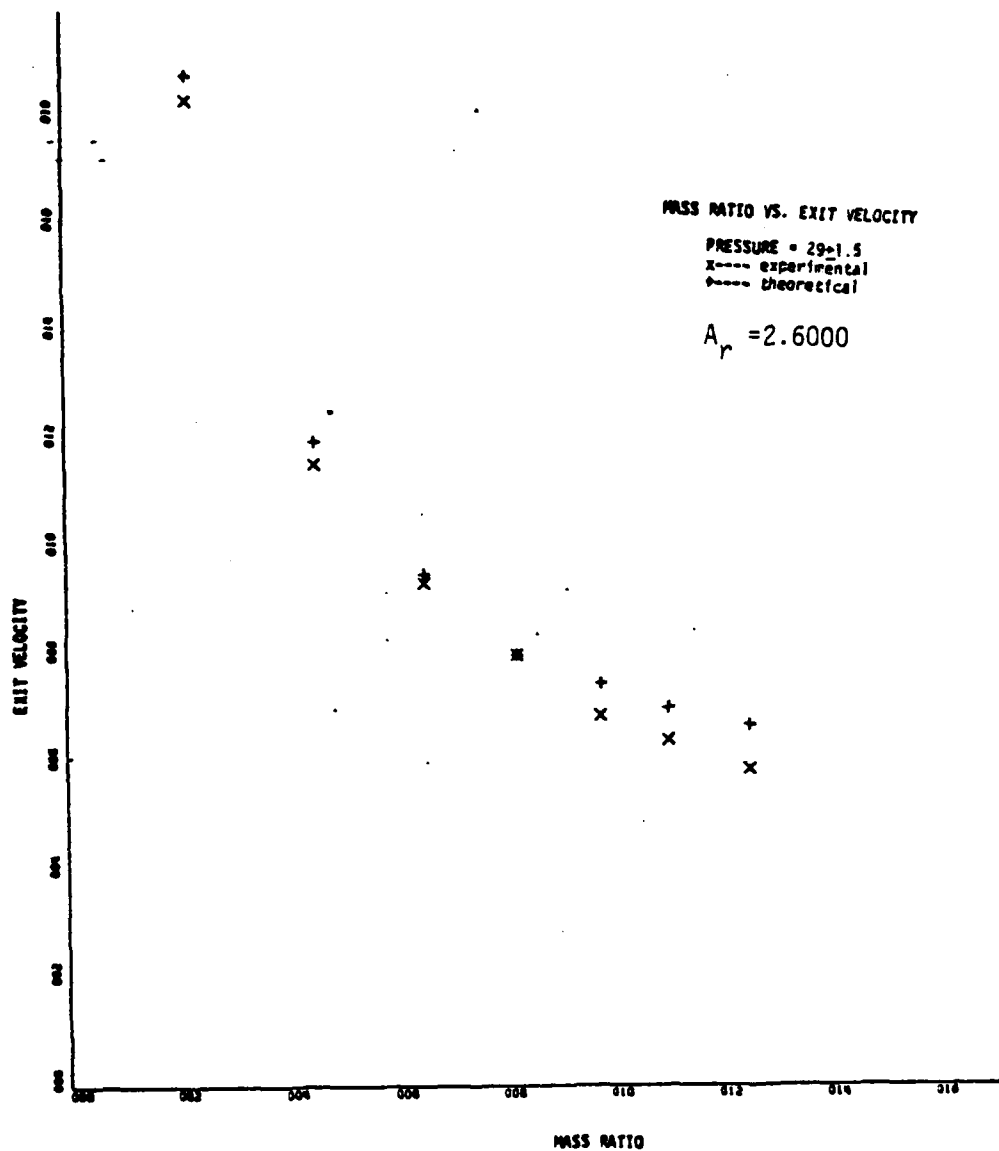
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=2.00E+02 UNITS INCH.

Figure 30. Mass Ratio vs. Exit Velocity at
 Pressure = 29±1.5psi Area = .84375



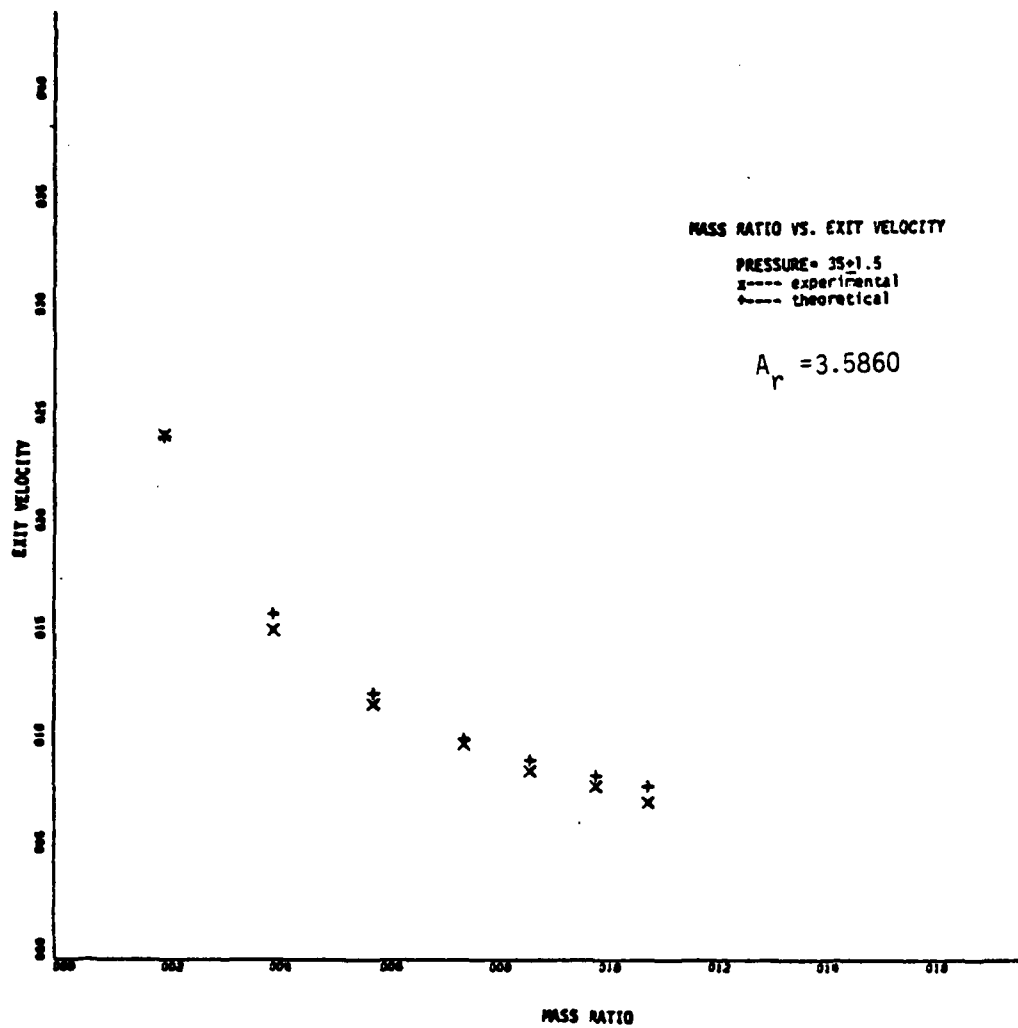
X-SCALE = $2.00E+00$ UNITS INCH.
Y-SCALE = $5.00E+02$ UNITS INCH.

Figure 31. Mass Ratio vs. Exit Velocity at
Pressure = 35 ± 1.5 psi Area = .62500



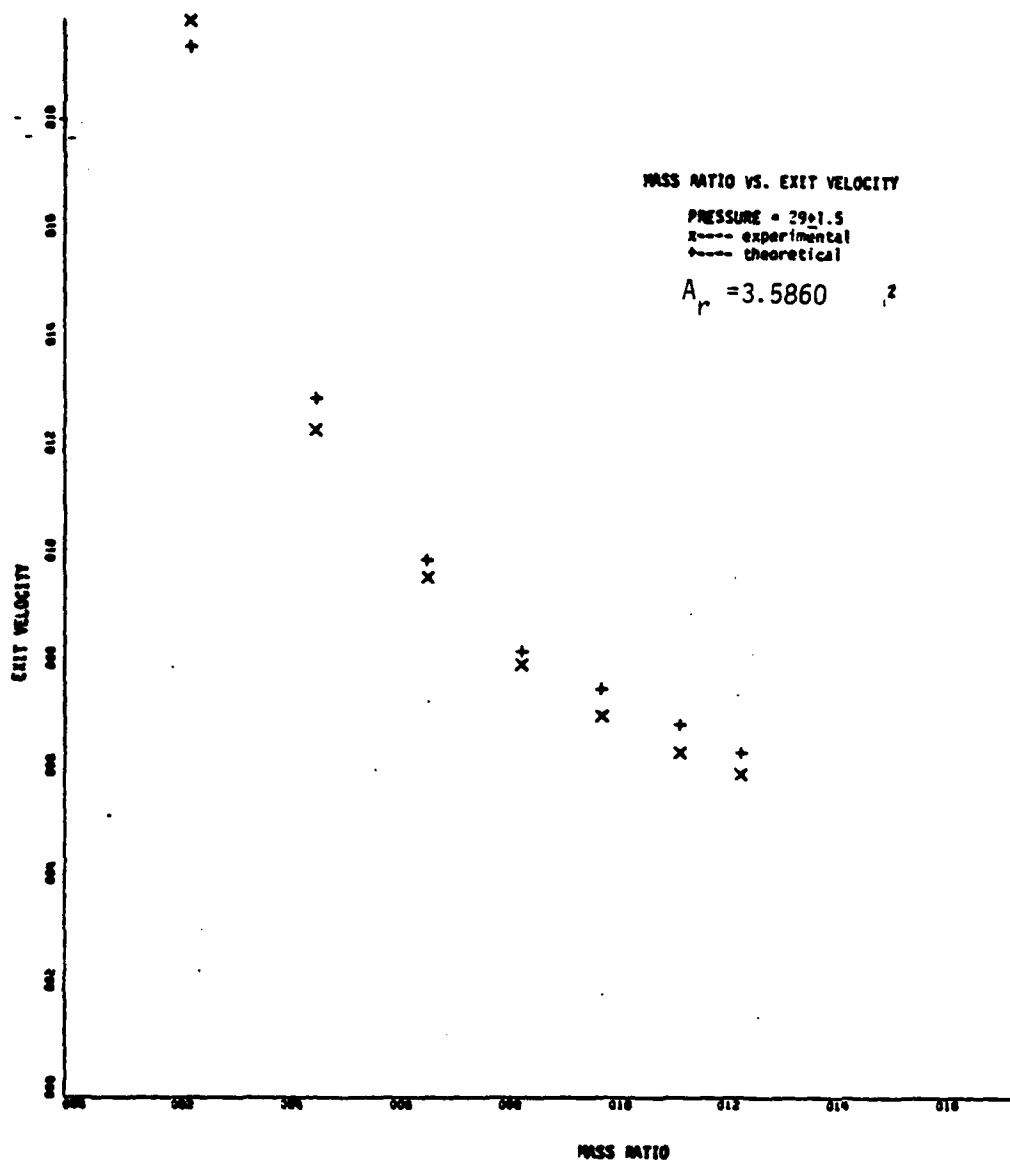
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 32. Mass Ratio vs. Exit Velocity at
Pressure = 29±1.5psi Area = .62500



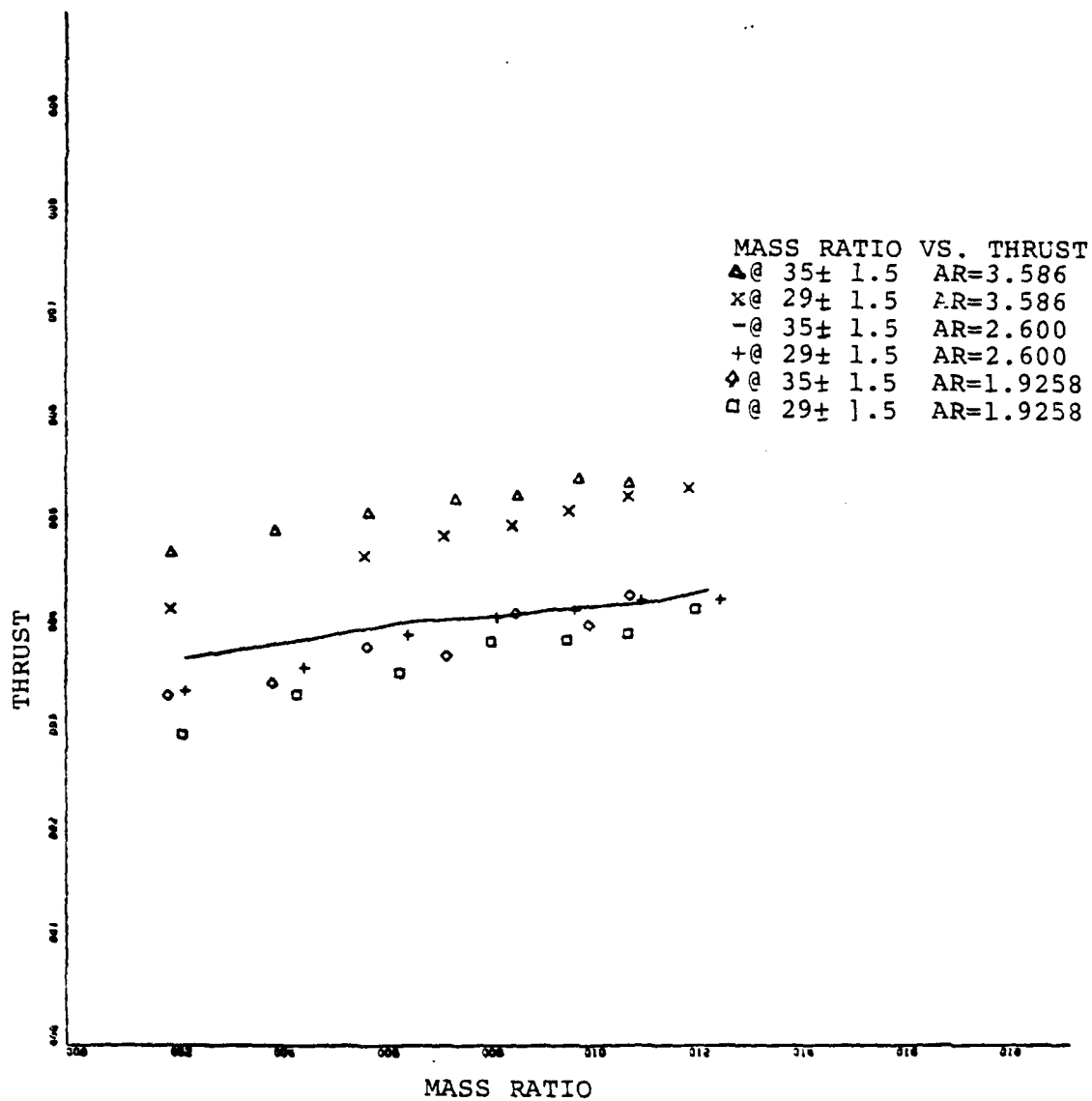
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=5.00E+02 UNITS INCH.

Figure 33. Mass Ratio vs. Exit Velocity at
Pressure = 35±1.5psi Area = .45313



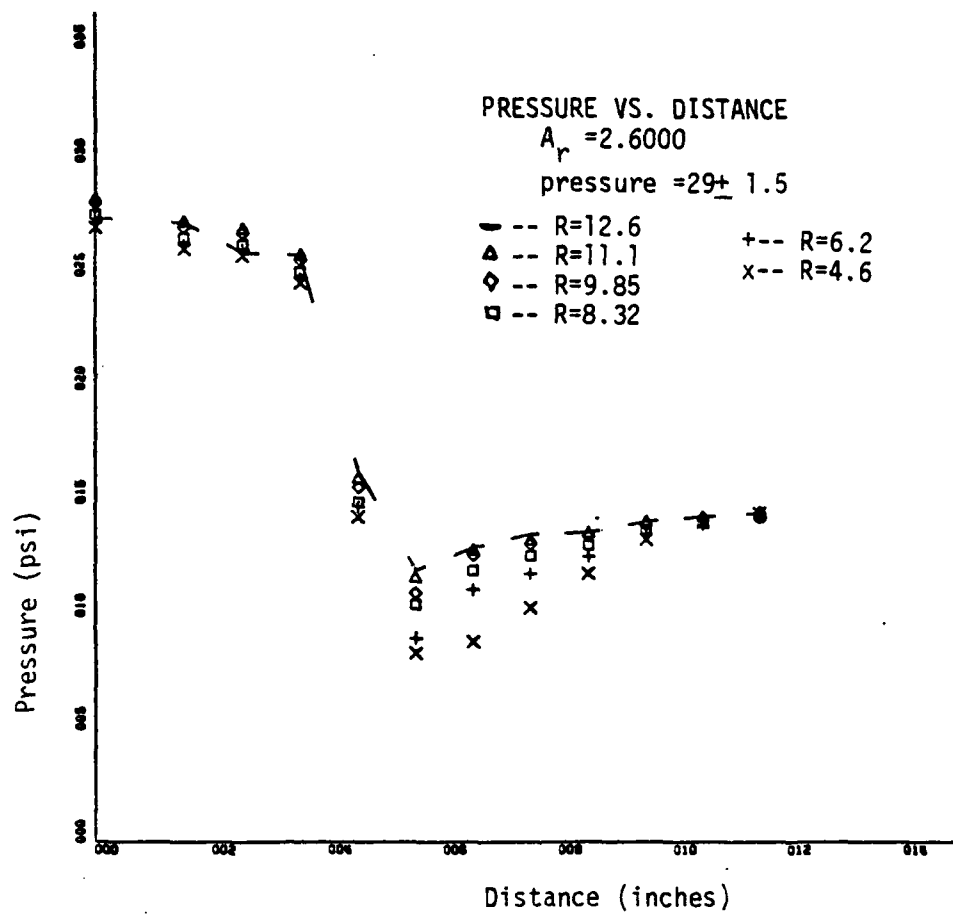
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 34. Mass Ratio vs. Exit Velocity at
Pressure = 29±1.5psi Area = .45313



X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=1.00E+00 UNITS INCH.

Figure 35. Mass Ratio vs. Thrust Curve



X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=5.00E+00 UNITS INCH.

Figure 36. Pressure vs. Distance at Pressure
 = 29 ± 1.5 Exit Area = .625 sq. in.

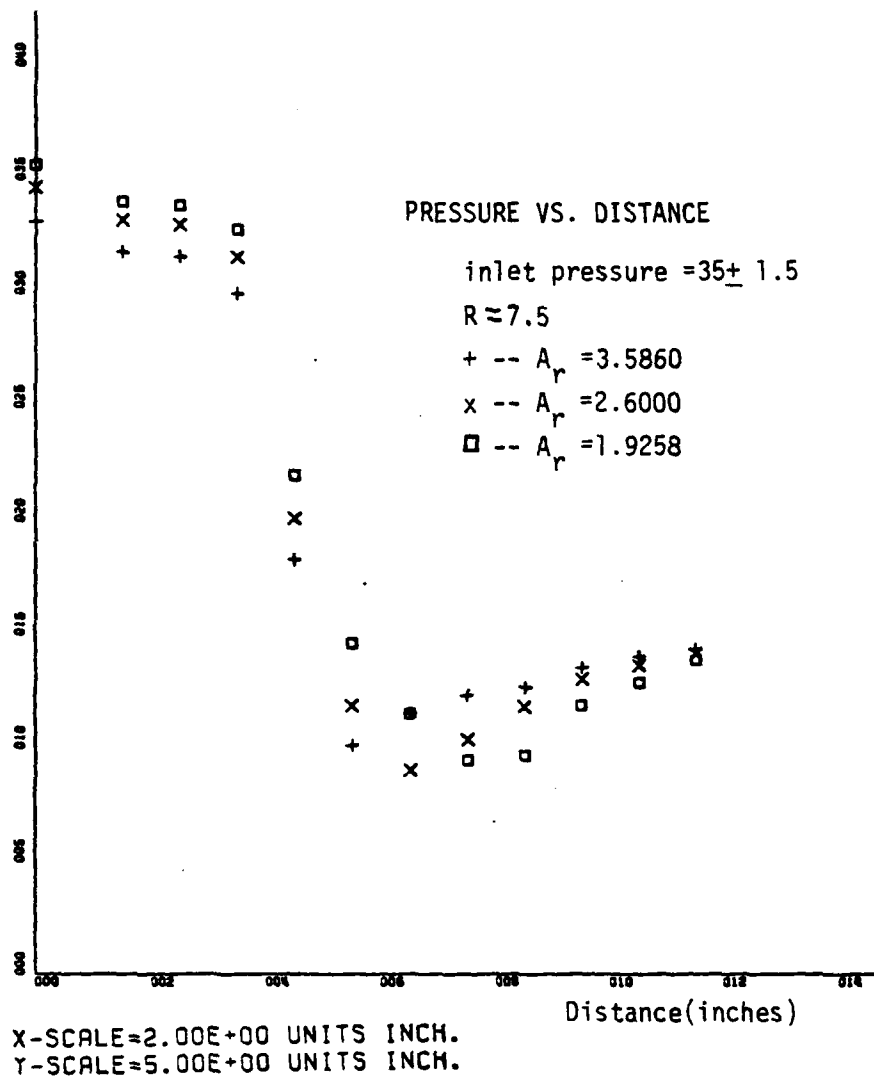


Figure 37. Pressure vs. Distance at
 $R \approx 7.5$ $p = 35\text{psi} \pm 1.5$

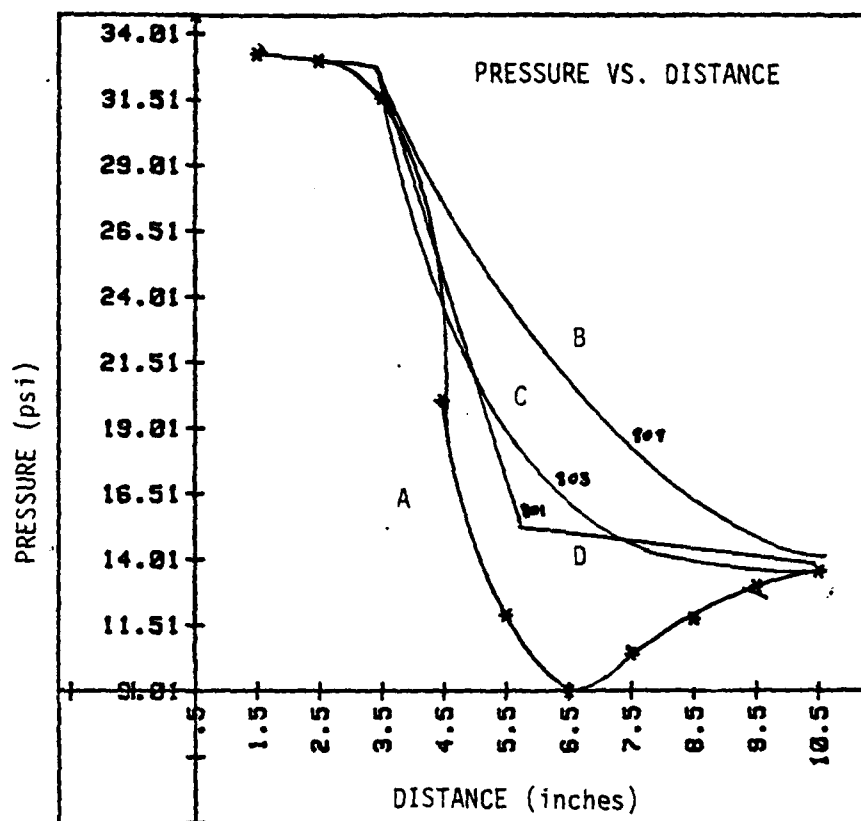


Figure 38. Various Pressure Profiles & Corresponding Exit Velocities

VIII. CONCLUSION

A set of experiments were conducted with an air-water mixture flowing at a mass ratio from 2 to 13. The experiments correlated well with a dual-phase two-component computer program. It appears that the program will permit prediction of the exit velocities to an accuracy of 10% for two-phase nozzles. At small nozzle area ratios, accuracies between theoretical and experimental are better; approximately 5%. All predicted velocities are higher than measured experimental values due to estimation of the drop size and the initial kinetic energy of the liquid at injection point.

It would be desirable to develop a drop size subroutine to better estimate varying drop size. The experimental system can be improved by velocity measurement devices at the inlet. Input of the gas and liquid velocities are necessary in the dual-phase two-component program and thus these measurement devices are vital for better accuracy.

Due to the insensitivity of the pressure distribution, it appears that any reasonable approximation to the pressure profile can be employed in the two-phase two-component flow program. For a given nozzle exit area and inlet pressure, a pressure distribution curve can be obtained through the experimental system. This distribution can therefore be used to obtain good results.

APPENDIX A: SAMPLE P(X) PROGRAM

```

$JOB
10  REAL P(75),X(75),PX,FX
11  INTEGER N,K,NP1,NX
12  A=35.2654923
13  B=5.6E5E777
14  C=14.39
15  D=0.04731
16  E=.444AC3
17  NX=75
18  P1=35.0
19  P2=8.0
20  PREF=8.0
21  XREF=10.0
22  N=74
23  NP1=N+1
24  PX=(P1-P2)/380
25  CC 10 K=1,N
26  X(K)=7.7*(K-1)/(N-1)
27  DX=7.7-X(K)
28  IF (X(K).LE.0.7) P(K)=P2+PX*E*(855-A*(X(K)+.5)**B)
29  IF (X(K).GT.0.7) P(K)=P2+PX*E*(C*DX+D*DX**4.9)
30  IF (X(K).EQ.1) GO TO 10
31  FRAC=(P(K-1)-PREF)/(P(K-1)-P(K))
32  IF (FRAC.GE.0.0.AND.FPAC.LE.1) XXREF=X(K-1)+FRAC*(X(K)-X(K-1))
33  10 CONTINUE
34  FX=XREF/XXREF
35  DC 15 K=1,N
36  X(K)=FX*X(K)
37  P(NP1)=1E5
38  CC 20 K=1,75.3
39  WRITE(6,100) P(K),X(K),P(K+1),X(K+1),P(K+2),X(K+2)
40  100 STOP
41  FORMAT(1E12.6)
42  END

```

[illegible]

```

45 IF(ABS(P(1)) - PRESS(K)) .GT. .001) GO TC 55
   X=C
   GC TC 60
55 CONTINUE
   IF ((P(1)) .GT. PRESS(K)) .AND. (P(1)) .LT. PRESS(X)) GC TC 60
   K=1+K
   X=X+1
   GC TC 55
60 CONTINUE
   IF ((C .NE. Q) .OR. (X .NE. O)) GO TC 61
   ANS(1)=TABLE(N,K)
   GC TC 90
61 CONTINUE
   IF (C .NE. C) GO TO 63
   D1 = TABLE(N,K) - TABLE(N,X)
   D7 = PRESS(K-1) - PRESS(X-1)
   D8 = PRESS(K-1) - P(1)
   D9 = (D7/D8)*D1
   ANS(1)=C+TABLE(N,K)
   GC TO 90
62 CONTINUE
   IF (X .NE. C) GO TO 70
   D1 = (TABLE(N,K) - TABLE(Q,K))
   D3 = TABLE(N,1) - TABLE(Q,1)
   D4 = TABLE(N,1) - TEMP(J)
   D5 = (D4/D3)*D1
   ANS(1)=C5+(TABLE(N,K))
   GC TC 90
70 CONTINUE
   D1 = TABLE(N,K) - TABLE(Q,K)
   D2 = TABLE(N,X) - TABLE(Q,X)
   D3 = TABLE(N,1) - TABLE(Q,1)
   D4 = TABLE(N,1) - TEMP(J)

```

[illegible]


```

3JCB      REAL PRFSS(9),TABLE(20,20),P(20),TEMP(20),ANS(20),D1,C2,D3,D4,C5,
          C6,D7,C8,C9,A,B,MWS(20)
          INTEGER I,J,I1,I2,K,X
          DATA PRFSS(1:9),A,B,MWS(1:20)/
          READ (5,100) ((TABLE(J,I),I=1,10),J=1,16)
          DO 10 N=1,5
          C5=PRFSS(N)=PRESS(N)*14.696
          CONTINUE
          DO 20 J=1,16
          C6=D7=C8=C9=A*B*MWS(20)
          I=2,10
          C3=TABLE(J,I)=TABLE(J,1)*.0807223
          CONTINUE
          C4=C5
          DO 52 J=1,16
          TEMP(J)=400.0+M
          A=M*10.0
          B=0.0
          DO 91 I=1,11
          P(I)=A
          N=1
          C=N+1
          A=100.0+A
          IF (RES(TEMP(J))-TABLE(N,1)).GT .001) GO TO 25
          C=C
          GO TO 50
          CONTINUE
          IF ((TEMP(J) .GT. TABLE(N,1)) .AND. (TEMP(J) .LT. TABLE(0,1)))
          1
          GO TO 50
          N=N+1
          C=N+1
          GO TO 15
          CONTINUE
          K=2
          X=K+1
          IF (P(1)).GT. .0001) GO TO 49
          X=C
          GO TO 60
          IF (ABS(P(1) -PRESS(K)).GT..01) GO TO 55
          X=C
          GO TO 60
          50
          20
          25
          15
          10
          5
          45

```

```

55      CONTINUE
      IF ((P(I) .GT. PRESS(K)) .AND. (F(I) .LT. PRESS(X))) GO TO 60
      K=I+K
      X=X+1
      GC TC 55
60      CONTINUE
      IF ((C.NE. 0) .OR. (X .NE. 0)) GO TO 61
      AN=(I)=TABLE(N,K)
      GC TC 50
61      CONTINUE
      IF ((C.NE. 0) GO TC 63
      D1 = (TABLE(N,K)-TABLE(N,X))
      D2 = (PRESS(K-1) -PRESS(X-1))
      D3 = PRESS(K-1)-P(I)
      D4 = ABS((C2/D3)*D1)
      AN(I)=D4+TABLE(N,K)
      GC TC 50
62      CONTINUE
      IF ((C.NE. 0) GO TO 70
      D1 = (TABLE(N,K)-TABLE(C,K))
      D2 = TABLE(N,1)-TABLE(C,1)
      D3 = TABLE(N,1)-TEMP(J)
      D4 = ABS((C2/D3)*D1)
      AN(I) = (TABLE(N,K))-D9
      GC TC 50
70      CONTINUE
      D1 = TABLE(N,K)-TABLE(C,K)
      D2 = TABLE(N,X)-TABLE(C,X)
      D3 = TABLE(N,1)-TABLE(C,1)
      D4 = TABLE(N,1)-TEMP(J)

```


APPENDIX D: SAMPLE PROPERTY TABLE

SENTPT							
400.0	.2494	100.0	.2534	200.0	.2571	CAG	0010
0.0	.2626	400.0	.2578	500.0	.2730	CAG	0020
300.0	.2779	700.0	.2642	800.0	.2767	CAG	0030
600.0	.3001	1000.0	.3053		1.ES	1.25CAG	0050
900.0						CAG	0060
410.0	.2488	100.0	.2532	200.0	.2569	CAG	0070
0.0	.2621	400.0	.2575	500.0	.2721	CAG	0080
300.0	.2773	700.0	.2643	800.0	.2718	CAG	0090
600.0	.2996	1000.0	.3030		1.ES	1.ES CAG	0100
900.0						CAG	0110
420.0	.2484	100.0	.2531	200.0	.2566	CAG	0120
0.0	.2616	400.0	.2575	500.0	.2712	CAG	0130
300.0	.2769	700.0	.2615	800.0	.2704	CAG	0140
600.0	.2990	1000.0	.3000		1.ES	1.ES CAG	0150
900.0						CAG	0160
430.0	.2484	100.0	.2529	200.0	.2564	CAG	0170
0.0	.2612	400.0	.2574	500.0	.2704	CAG	0180
300.0	.2768	700.0	.2611	800.0	.2709	CAG	0190
600.0	.2924	1000.0	.2970		1.ES	1.ES CAG	0200
900.0						CAG	0210
440.0	.2484	100.0	.2527	200.0	.2561	CAG	0220
0.0	.2607	400.0	.2551	500.0	.2695	CAG	0230
300.0	.2737	700.0	.2707	800.0	.2647	CAG	0240
600.0	.2898	1000.0	.2942		1.ES	1.ES CAG	0250
900.0						CAG	0260
450.0	.2484	100.0	.2526	200.0	.2559	CAG	0270
0.0	.2602	400.0	.2545	500.0	.2636	CAG	0280
300.0	.2726	700.0	.2718	800.0	.2625	CAG	0290
600.0	.2872	1000.0	.2915		1.ES	1.ES CAG	0300
900.0						CAG	0310
460.0	.2484	100.0	.2524	200.0	.2556	CAG	0320
0.0	.2597	400.0	.2618	500.0	.2678	CAG	0330
300.0	.2716	700.0	.2759	800.0	.2604	CAG	0340
600.0							

[illegible]

307.0	28.508	400.0	28.710	500.0	28.942	WA	1070
600.0	29.055	700.0	29.242	400.0	29.331	WA	1080
900.0	29.460	1000.0	29.500		1.25	1.25	1090
0.0							1100
0.0	28.010	100.0	28.175	200.0	28.310	WA	1110
300.0	28.472	400.0	28.453	500.0	28.851	WA	1120
600.0	28.958	700.0	29.127	400.0	29.200	WA	1130
900.0	29.312	1000.0	29.425		1.25	1.25	1140
0.0							1150
0.0	28.010	100.0	28.153	200.0	28.294	WA	1160
300.0	28.436	400.0	28.725	500.0	28.765	WA	1170
600.0	28.863	700.0	29.012	400.0	29.073	WA	1180
900.0	29.165	1000.0	29.255		1.25	1.25	1190
0.0							1200
0.0	28.010	100.0	28.151	200.0	28.266	WA	1210
300.0	28.403	400.0	28.743	500.0	28.980	WA	1220
600.0	29.768	700.0	29.032	400.0	29.940	WA	1230
900.0	29.020	1000.0	29.310		1.25	1.25	1240
0.0							1250
0.0	28.010	100.0	28.140	200.0	28.244	WA	1260
300.0	28.383	400.0	28.682	500.0	28.945	WA	1270
600.0	28.677	700.0	28.950	400.0	29.017	WA	1280
900.0	29.899	1000.0	29.340		1.25	1.25	1290
0.0							1300
0.0	28.010	100.0	28.123	200.0	28.222	WA	1310
300.0	28.127	400.0	28.422	500.0	28.517	WA	1320
600.0	28.547	700.0	28.748	400.0	28.700	WA	1330
900.0	28.750	1000.0	28.730		1.25	1.25	1340
0.0							1350
0.0	28.010E+01	100000E+02	2.41160E+02	270000E+01	2.42100E+01	WA	1360
300.0	2.41100E+01	400000E+01	2.41750E+02	500000E+01	2.44400E+02	WA	1370
600.0	2.45740E+02	700000E+01	2.45500E+02	400000E+01	2.45100E+02	WA	1380
900.0	2.46270E+02	1000000E+04	2.46190E+02	1000000E+06	1.00000E+06	WA	1390
0.0							1400
0.0	2.40110E+02	1000000E+03	2.41240E+02	2000000E+01	2.41000E+02	WA	1410
300.0	2.42530E+02	4000000E+01	2.41100E+02	5000000E+01	2.41730E+02	WA	1420
600.0	2.44240E+02	7000000E+01	2.44400E+02	4000000E+01	2.44000E+02	WA	1430
900.0	2.45110E+02	10000000E+04	2.45180E+02	10000000E+06	1.00000E+06	WA	1440
0.0							1450
0.0	2.40100E+02	10000000E+03	2.40920E+02	20000000E+01	2.41550E+02	WA	1460
300.0	2.41100E+02	40000000E+01	2.41400E+02	50000000E+01	2.41100E+02	WA	1470
600.0	2.41500E+02	70000000E+01	2.41550E+02	40000000E+01	2.41100E+02	WA	1480
900.0	2.44120E+02	100000000E+04	2.44240E+02	100000000E+06	1.00000E+06	WA	1490
0.0							1500
0.0	2.40100E+02	100000000E+03	2.40400E+02	200000000E+01	2.41300E+02	WA	1510
300.0	2.41440E+02	400000000E+01	2.41200E+02	500000000E+01	2.42530E+02	WA	1520
600.0	2.42200E+02	700000000E+01	2.42400E+02	400000000E+01	2.41130E+02	WA	1530
900.0	2.42100E+02	1000000000E+04	2.41300E+02	1000000000E+06	1.00000E+06	WA	1540
0.0							1550
0.0	2.40110E+02	1000000000E+03	2.41640E+02	2000000000E+01	2.41120E+02	WA	1560
300.0	2.41500E+02	4000000000E+01	2.41400E+02	5000000000E+01	2.42200E+02	WA	1570
600.0	2.42200E+02	7000000000E+01	2.42400E+02	4000000000E+01	2.41150E+02	WA	1580
900.0	2.42150E+02	10000000000E+04	2.41350E+02	10000000000E+06	1.00000E+06	WA	1590
0.0							1600
0.0	2.40110E+02	10000000000E+03	2.41540E+02	20000000000E+01	2.41200E+02	WA	1610
300.0	2.41200E+02	40000000000E+01	2.41410E+02	50000000000E+01	2.41500E+02	WA	1620
600.0	2.41540E+02	70000000000E+01	2.41480E+02	40000000000E+01	2.41600E+02	WA	1630
900.0	2.41560E+02	100000000000E+04	2.41480E+02	100000000000E+06	1.00000E+06	WA	1640
0.0							1650
0.0	2.41100E+02	100000000000E+03	2.41480E+02	200000000000E+01	2.40700E+02	WA	1660
300.0	2.40920E+02	400000000000E+01	2.41180E+02	500000000000E+01	2.41240E+02	WA	1670
600.0	2.41120E+02	700000000000E+01	2.41100E+02	400000000000E+01	2.41900E+02	WA	1680
900.0	2.40400E+02	1000000000000E+04	2.41540E+02	1000000000000E+06	1.00000E+06	WA	1690
0.0							1700
0.0	2.40100E+02	1000000000000E+03	2.41400E+02	2000000000000E+01	2.40550E+02	WA	1710
300.0	2.40640E+02	4000000000000E+01	2.41750E+02	5000000000000E+01	2.40730E+02	WA	1720
600.0	2.40670E+02	7000000000000E+01	2.41540E+02	4000000000000E+01	2.40150E+02	WA	1730
900.0	2.40150E+02	10000000000000E+04	2.79920E+02	10000000000000E+06	1.00000E+06	WA	1740
0.0							1750
0.0	2.40000E+02	10000000000000E+03	2.41100E+02	20000000000000E+01	2.40440E+02	WA	1760
300.0	2.40000E+02	40000000000000E+01	2.40400E+02	50000000000000E+01	2.40160E+02	WA	1770
600.0	2.40260E+02	70000000000000E+01	2.40000E+02	40000000000000E+01	2.79840E+02	WA	1780

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APPENDIX E: SAMPLE INPUT DATA

APRIL 15, 1980		NOZZLE D, 2 MPA		920		1	
1	0	30	30	30	30	30	30
-0.3	20.0	0.0	0.0	0.0	0.0	5.28	208.0
1010.0	85E-60.0	-0.3	28.02	18.02	20.0	537.0	0.1
20.0	0.05	290.0	537.0	537.0	0.003	0.003	
10	0.005	0.005	0.0034	0.0034			
IN. NCZLL F							
2500000	000000	289564+03	138144+00	288590+03	276288+00		
284616+03	414332+00	283025+03	552576+00	276898+03	690720+00		
266495+03	828864+00	252206+03	967008+00	236879+03	110515+01		
222415+03	124330+01	209777+03	138144+01	195929+03	151958+01		
183834+03	165773+01	172458+03	179537+01	161766+03	193402+01		
151728+03	207216+01	142310+03	221030+01	133482+03	234845+01		
125216+03	249655+01	117431+03	262474+01	110251+03	278289+01		
103500+03	290102+01	972006+02	303917+01	913291+02	317771+01		
858612+02	321546+01	807742+02	345360+01	760066+02	359174+01		
716545+02	372339+01	675910+02	396803+01	638066+02	400618+01		
613065+02	414432+01	570700+02	423246+01	540765+02	442061+01		
513102+02	455875+01	467553+02	469690+01	463268+02	482504+01		
442206+02	497315+01	422129+02	511133+01	403611+02	524947+01		
380527+02	533762+01	370764+02	552576+01	356211+02	565351+01		
342765+02	560202+01	329228+02	594019+01	318810+02	597831+01		
308124+02	621648+01	298120+02	635463+01	264540+02	649777+01		
280228+02	663091+01	272169+02	676906+01	244000+02	690720+01		
257235+02	705535+01	250505+02	713249+01	226035+02	732123+01		
237779+02	745078+01	231803+02	759702+01	206684+02	773607+01		
220445+02	787421+01	215603+02	801235+01	209257+02	815050+01		
204466+02	828864+01	193329+02	842679+01	179296+02	856453+01		
185235+02	870307+01	184252+02	884122+01	164525+02	897936+01		
174360+02	911751+01	169438+02	925565+01	149807+02	939370+01		
159617+02	953194+01	154712+02	967008+01	100000+06	930823+01		
144904+02	994637+01	140000+02	100845+02		100000+06		

APPENDIX F: SAMPLE 140PC CALIBRATION PROGRAM

```

10 DIM X(15,100),Sum(15),Mean(15),Sum2(15),Sd(15)
30 N=10
40 FOR I=1 TO N
50   OUTPUT 709;"AI1VT1"
60   ENTER 709;X(1,I)
70   OUTPUT 709;"AI2VT1"
80   ENTER 709;X(2,I)
90   OUTPUT 709;"AI3VT1"
100  ENTER 709;X(3,I)
110  OUTPUT 709;"AI4VT1"
120  ENTER 709;X(4,I)
130  OUTPUT 709;"AI5VT1"
140  ENTER 709;X(5,I)
150  OUTPUT 709;"AI6VT1"
160  ENTER 709;X(6,I)
170  OUTPUT 709;"AI7VT1"
180  ENTER 709;X(7,I)
190  OUTPUT 709;"AI8VT1"
200  ENTER 709;X(8,I)
210  OUTPUT 709;"AI9VT1"
220  ENTER 709;X(9,I)
230  OUTPUT 709;"AI16VT1"
240  ENTER 709;X(10,I)
250  OUTPUT 709;"AI11VT1"
260  ENTER 709;X(11,I)
261  OUTPUT 709;"AI12VT1"
262  ENTER 709;X(12,I)
263  OUTPUT 709;"AI13VT1"
264  ENTER 709;X(13,I)
267  OUTPUT 709;"AI17VT1"
268  ENTER 709;X(14,I)
270 NEXT I

```

```

280 PRINT "
290 PRINT "
300 I
310 PRINT "
320 PRINT
330 FOR I=1 TO N
340 PRINT USING "DD.DDDD";X(1,I),X(2,I),X(3,I),X(4,I),X(5,I),X(6,I),X(7,I),
X(8,I)
350 NEXT I
360 PRINT
370 PRINT "
380 PRINT "
390 PRINT "
400 FOR I=1 TO N
410 PRINT USING "DD.DDDD";X(9,I),X(10,I),X(11,I),X(12,I),X(13,I),X(14,I)
411 NEXT I
413 PRINT
414 PRINT "
415 PRINT "
420 PRINT
440 FOR J=1 TO 14
450 Sum(J)=0
460 FOR I=1 TO N
470 Sum(J)=Sum(J)+X(J,I)
480 NEXT I
490 Mean(J)=Sum(J)/N
510 NEXT J
512 FOR J=1 TO 14
513 FOR I=1 TO N
515 Sum2(J)=Sum2(J)+X(J,I)-Mean(J))*(X(J,I)-Mean(J))
516 NEXT I
518 Sd(J)=SQR(Sum2(J)/(N-1))
519 IF J=14 THEN GOTO 523
521 PRINT "channel
522 GOTO 525
523 PRINT "CHANNEL
525 NEXT J
526 END

```

CHANNELS							
1	2	3	4	5	6	7	8
CHANNEL 9 10 11 12 13 17 MEAN " " STANDARD DEVIATION "							

APPENDIX C: SAMPLE OUTPUT TO CALIBRATION PROGRAM

THIS DATA IS FOR A PRESSURE OF 0 PSI

CHANNELS							
1	2	3	4	5	6	7	8
1.6588	1.8301	1.7702	1.6914	1.8429	1.7485	1.7900	1.8098
1.6589	1.8305	1.7705	1.6922	1.8431	1.7484	1.7901	1.8101
1.6598	1.8308	1.7715	1.6931	1.8446	1.7500	1.7918	1.8120
1.6605	1.8318	1.7717	1.6935	1.8444	1.7499	1.7916	1.8117
1.6603	1.8317	1.7722	1.6931	1.8446	1.7497	1.7917	1.8117
1.6598	1.8315	1.7716	1.6928	1.8439	1.7495	1.7909	1.8112
1.6596	1.8310	1.7712	1.6925	1.8437	1.7494	1.7909	1.8110
1.6597	1.8310	1.7714	1.6925	1.8441	1.7494	1.7910	1.8112
1.6593	1.8311	1.7712	1.6923	1.8438	1.7496	1.7907	1.8114
1.6594	1.8313	1.7709	1.6927	1.8438	1.7496	1.7906	1.8109

CHANNEL				
9	10	11	12	13
1.7941	1.7381	1.7211	1.8335	-.0119
1.7946	1.7389	1.7221	1.8336	-.0119
1.7961	1.7397	1.7239	1.8353	-.0119
1.7958	1.7400	1.7232	1.8341	-.0119
1.7960	1.7401	1.7237	1.8351	-.0119
1.7957	1.7402	1.7236	1.8347	-.0119
1.7955	1.7388	1.7233	1.8340	-.0119
1.7959	1.7394	1.7226	1.8342	-.0119
1.7958	1.7398	1.7233	1.8338	-.0119
1.7954	1.7394	1.7231	1.8341	-.0119

	MEAN	STANDARD DEVIATION
channel 1	1.65961	.000542524961023
channel 2	1.83108	.000528730134904
channel 3	1.77124	.000587272414548
channel 4	1.69261	.000583952052826
channel 5	1.84389	.000574359546703
channel 6	1.7494	.000537483849887
channel 7	1.79093	.0006254775953
channel 8	1.8111	.000697614984549
channel 9	1.79549	.000647130417905
channel 10	1.73944	.000668663675633
channel 11	1.72299	.000847807630172
channel 12	1.83424	.000607728009338
channel 13	-.0118729	8.1846740246E-6
channel 17	14.9017	.00368329562575

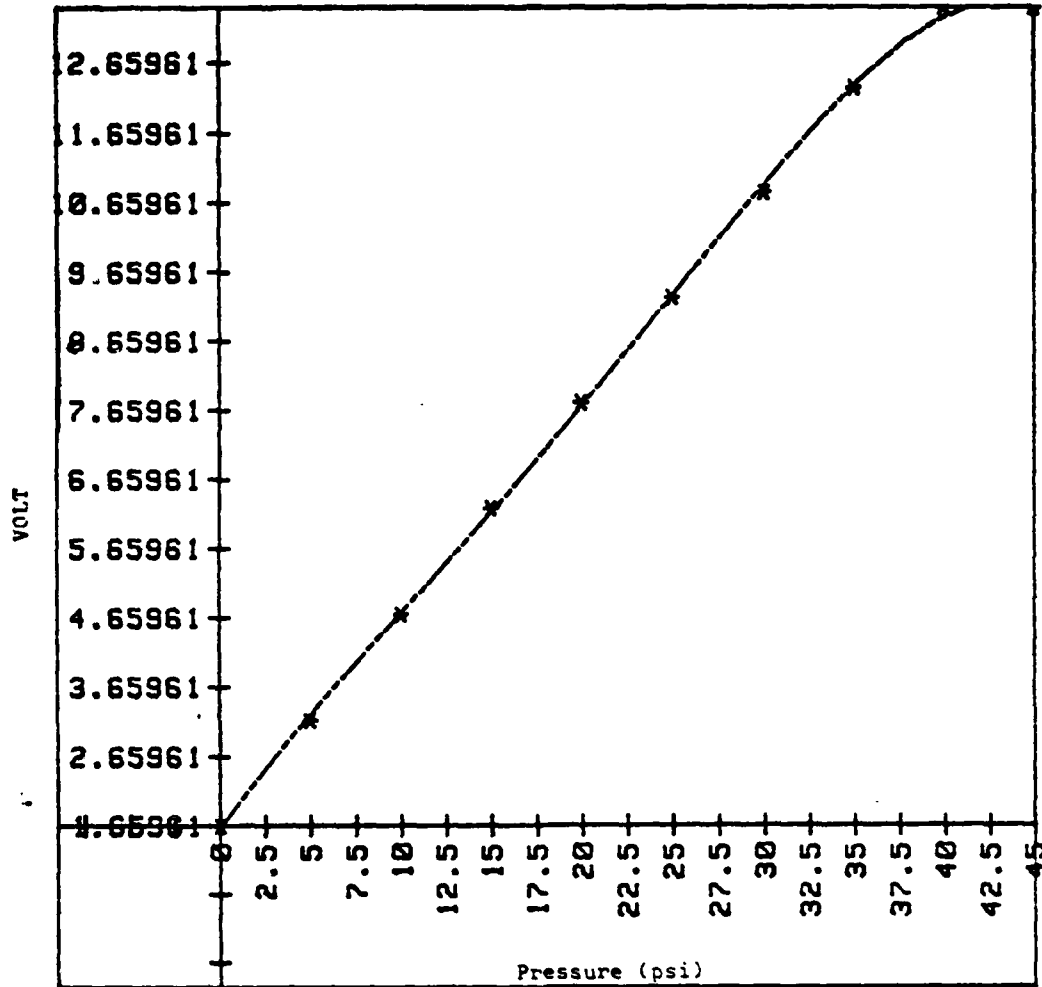
APPENDIX H: SAMPLE 200PC CALIBRATION PROGRAM

```

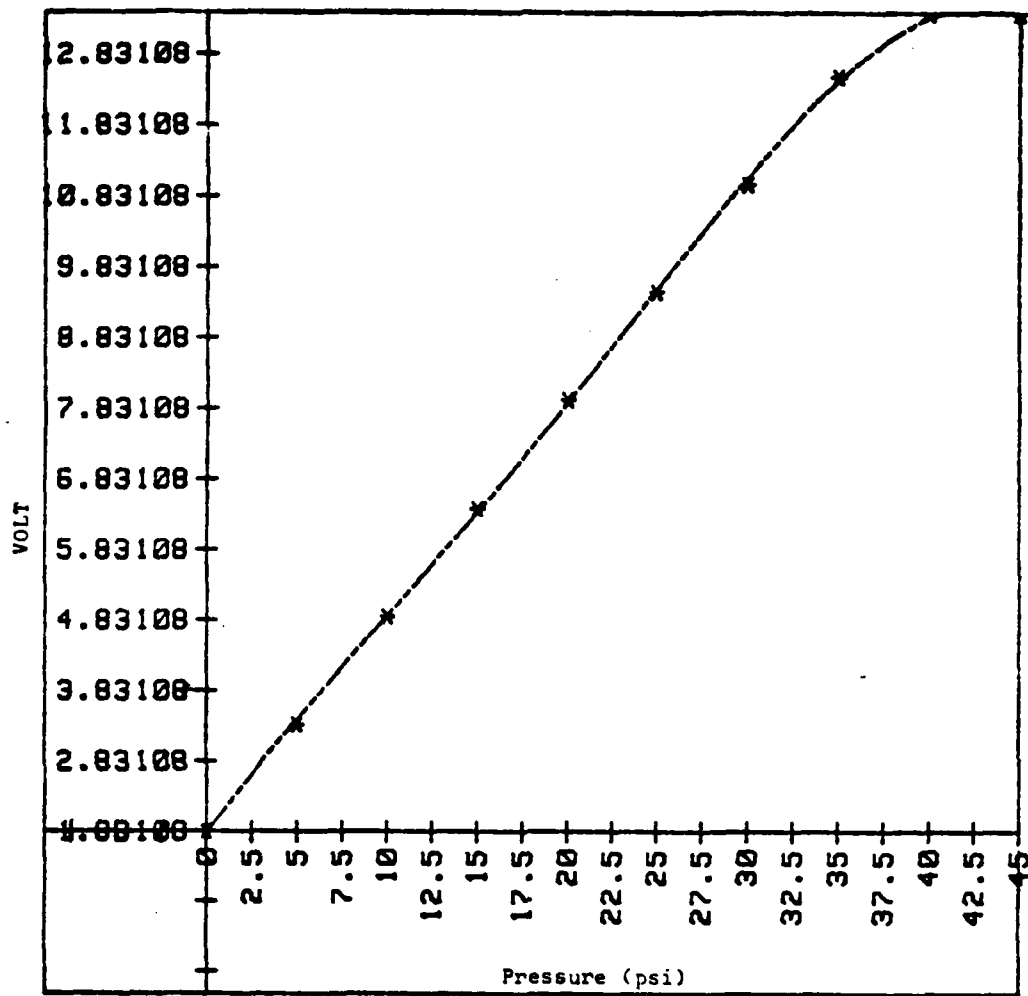
PRINT "THIS PROGRAM IS USED TO CALIBRATE 100 PSI PRESSURE TRANSDUCERS."
PRINT "ENTER THE AMOUNT OF READINGS YOU WISH THE PROGRAM TO AVERAGE"
DIM X(11,100),Sum(11),Mean(11),Sum2(11),Sd(11)
PRINT
INPUT N
Y=0
FOR I=1 TO 11
  PRINT
  PRINT
  PRINT "THIS RUN IS FOR A PRESSURE OF";Y;"PSI"
  PRINT
  PRINT
  PRINT "ONCE THE PRESSURE IS SET AT";Y;"PSI"
  PRINT "PUSH THE CONTINUE BUTTON AND WAIT FOR DATA."
  Y=Y+10
  PAUSE
  FOR J=1 TO N
    OUTPUT 709;"AI12VT11"
    ENTER 709;X(1,I)
    OUTPUT 709;"AI14VT11"
    ENTER 709;X(2,I)
    OUTPUT 709;"AI15VT11"
    ENTER 709;X(3,I)
  NEXT J

```

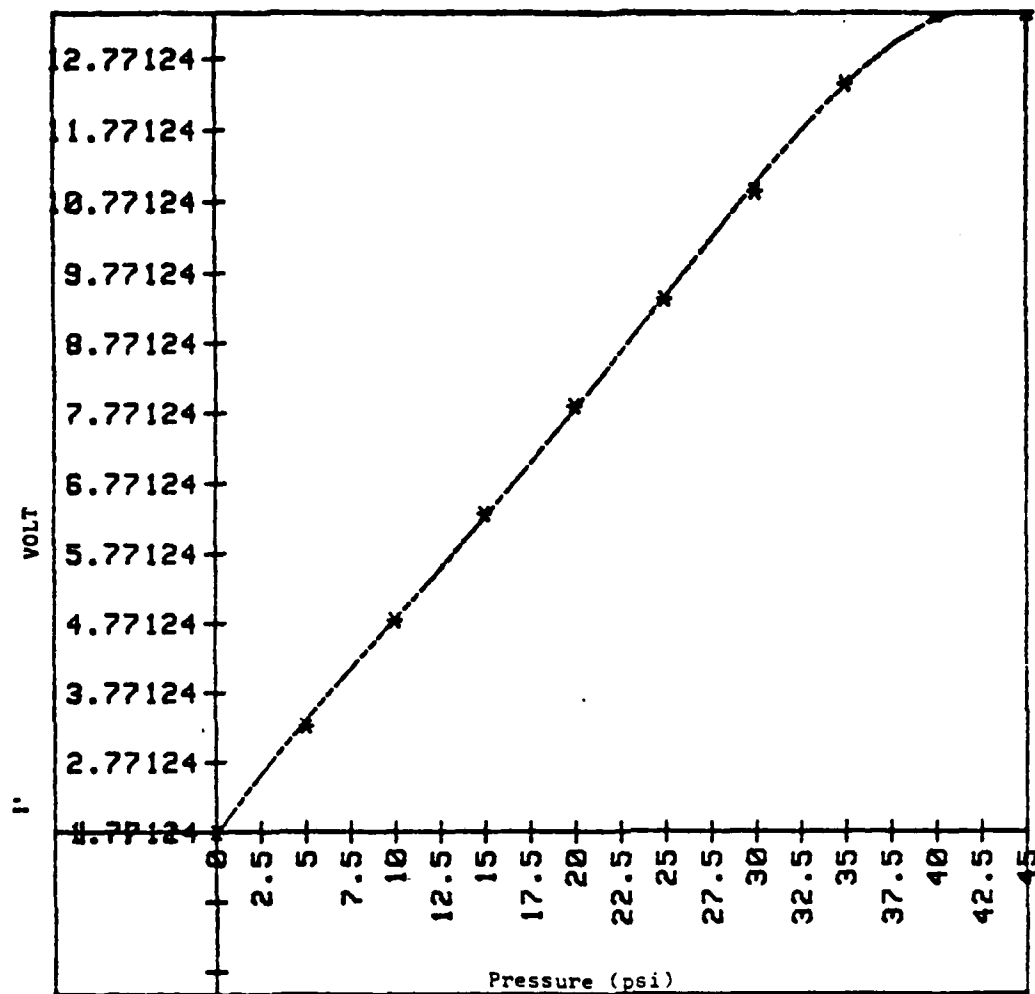

APPENDIX I: CALIBRATION PLOT OF PRESSURE TRANSDUCERS



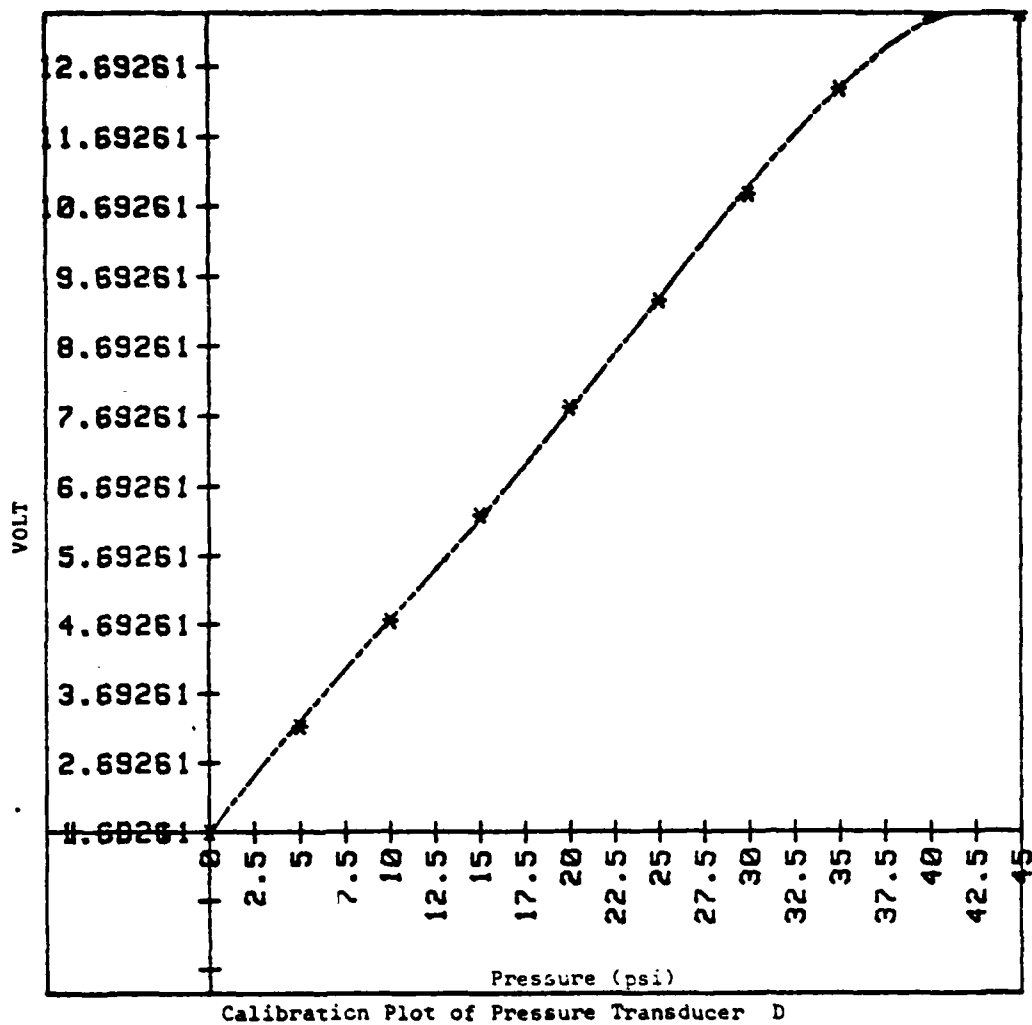
Calibration Plot of Pressure Transducer A

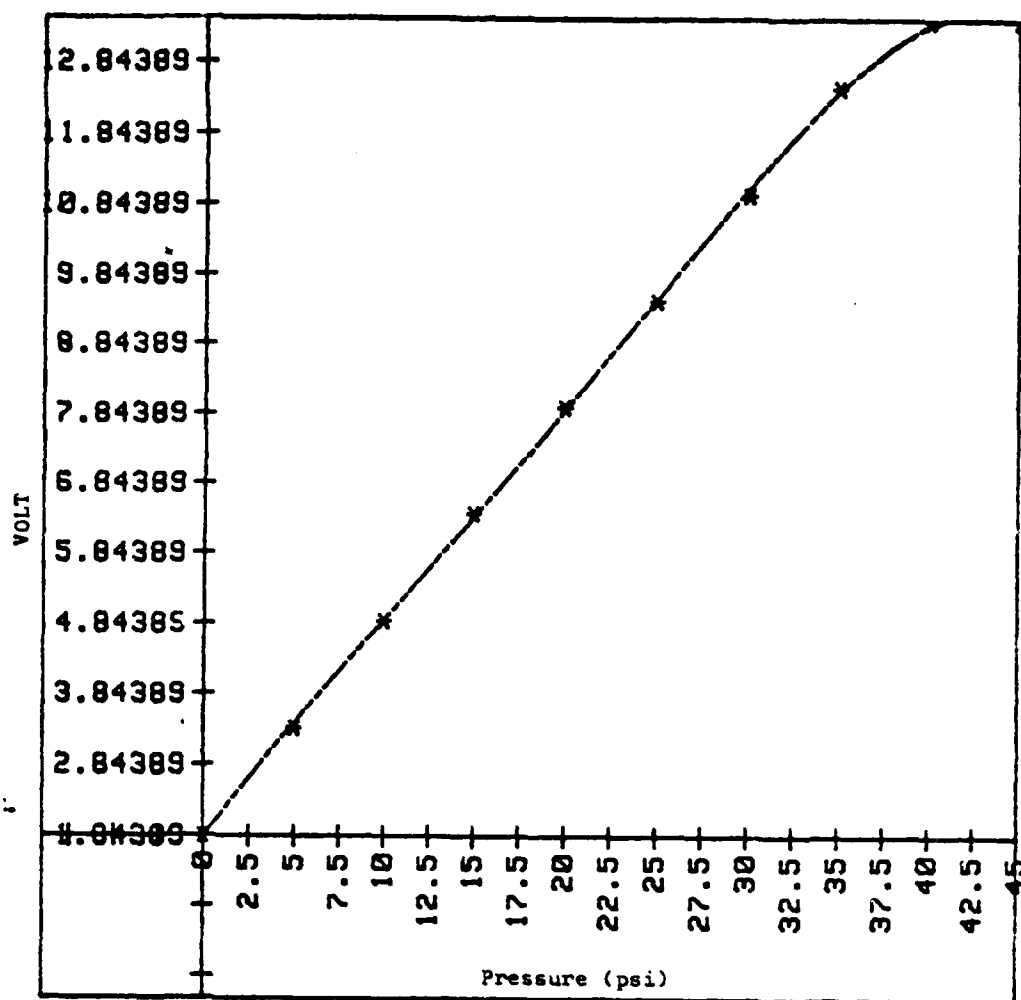


Calibration Plot of Pressure Transducer B

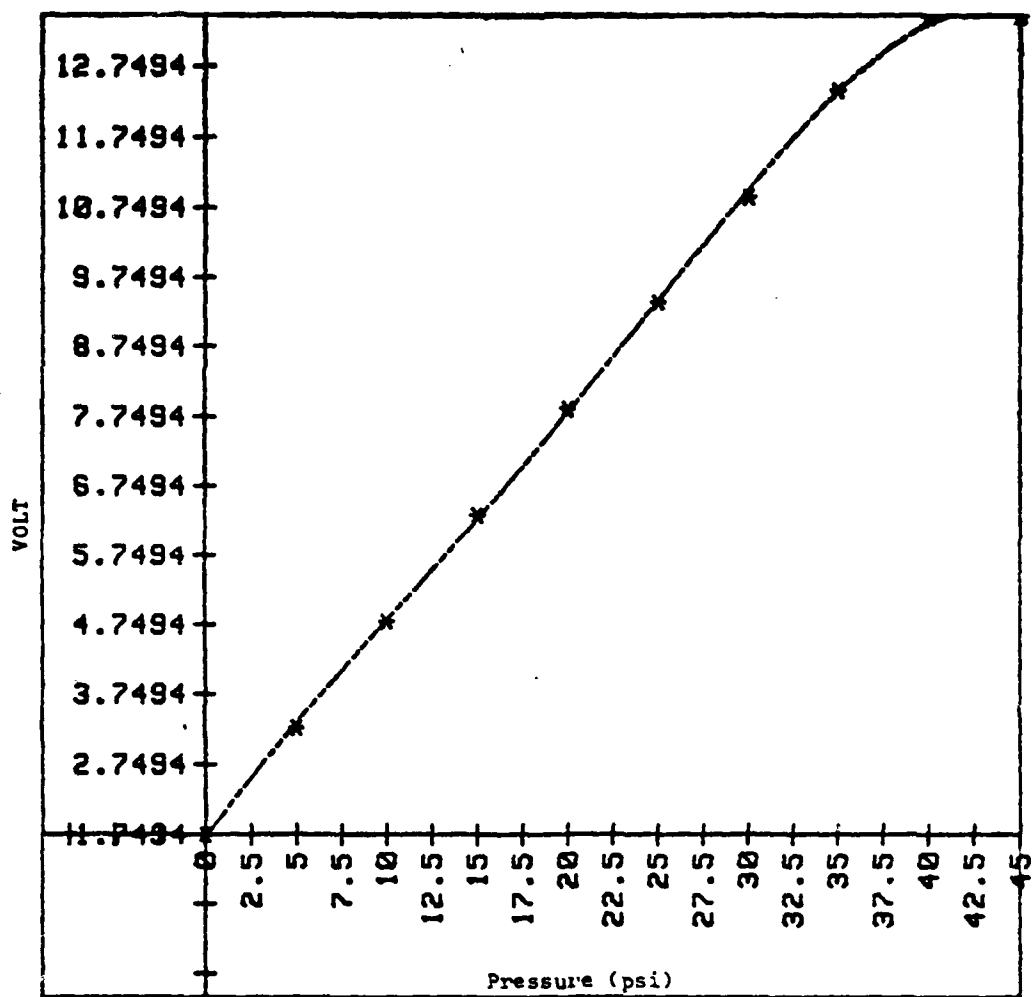


Calibration Plot of Pressure Transducer C

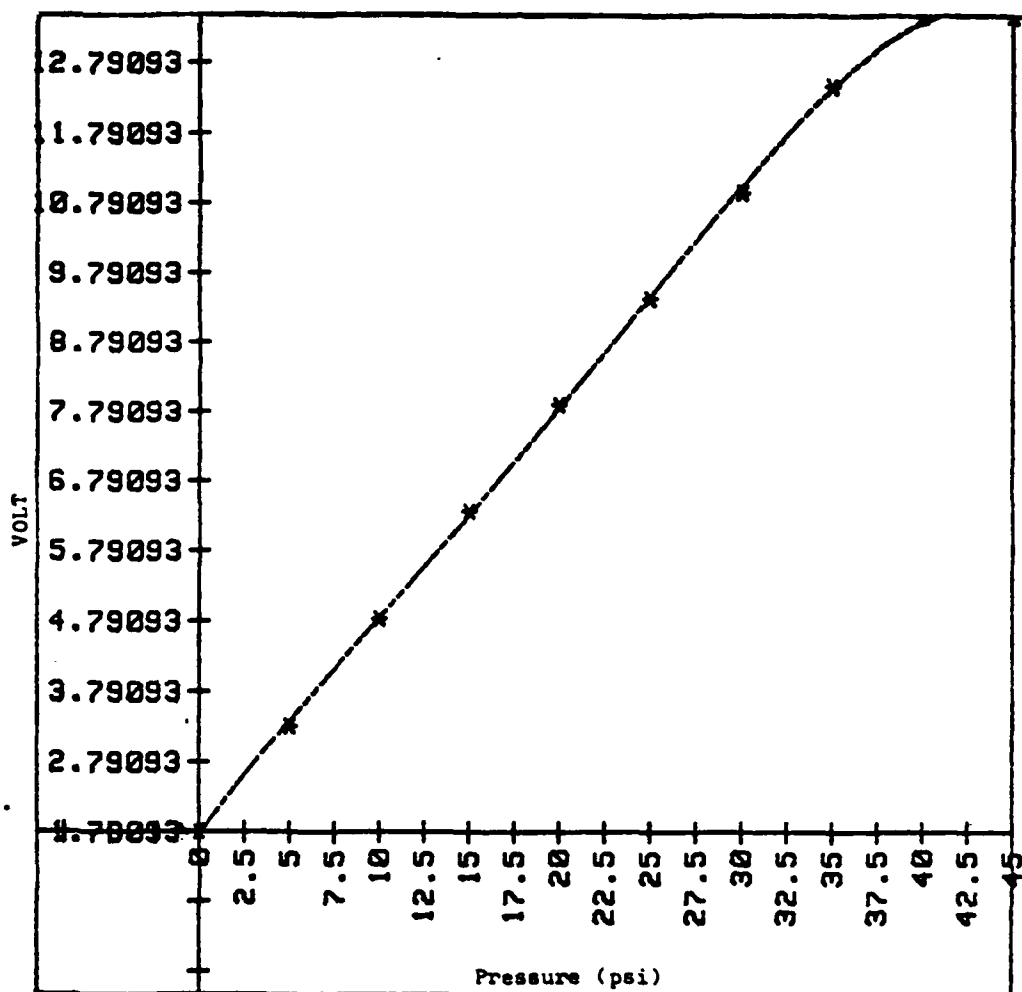




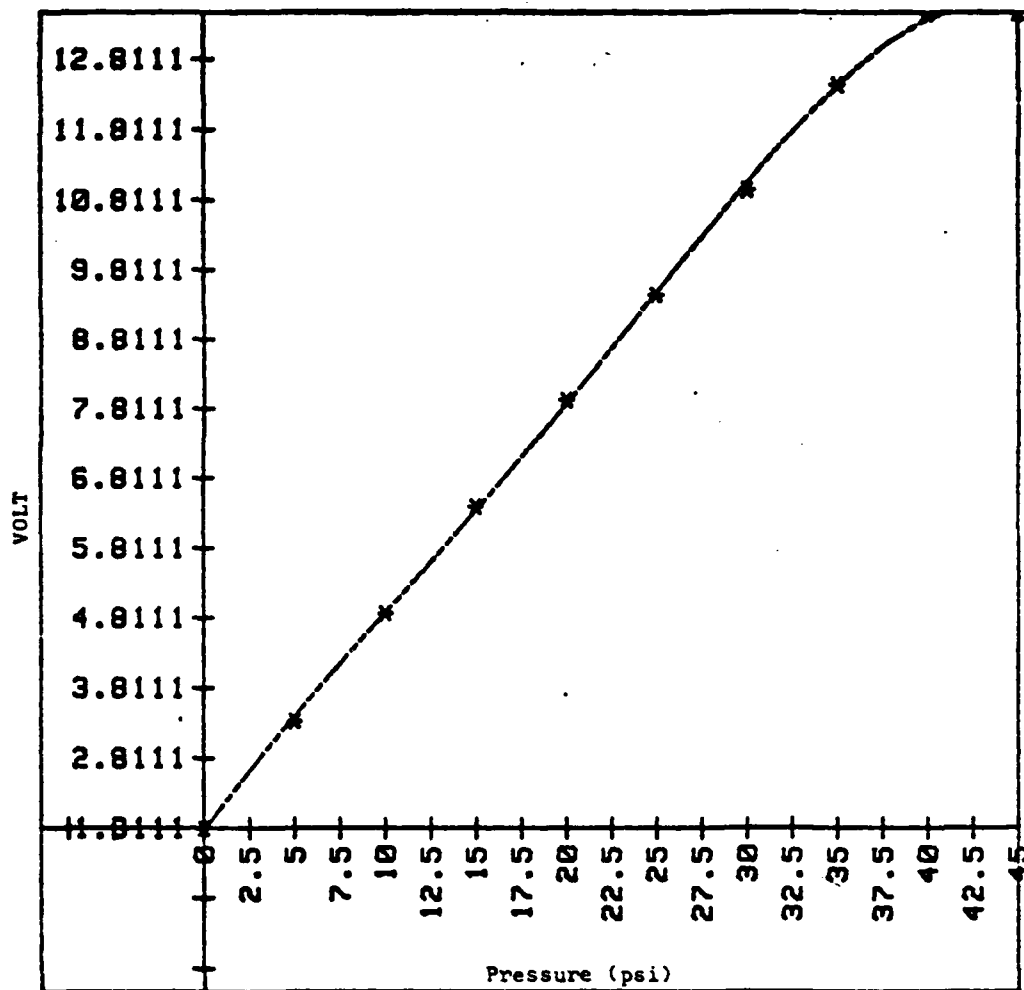
Calibration Plot of Pressure Transducer E



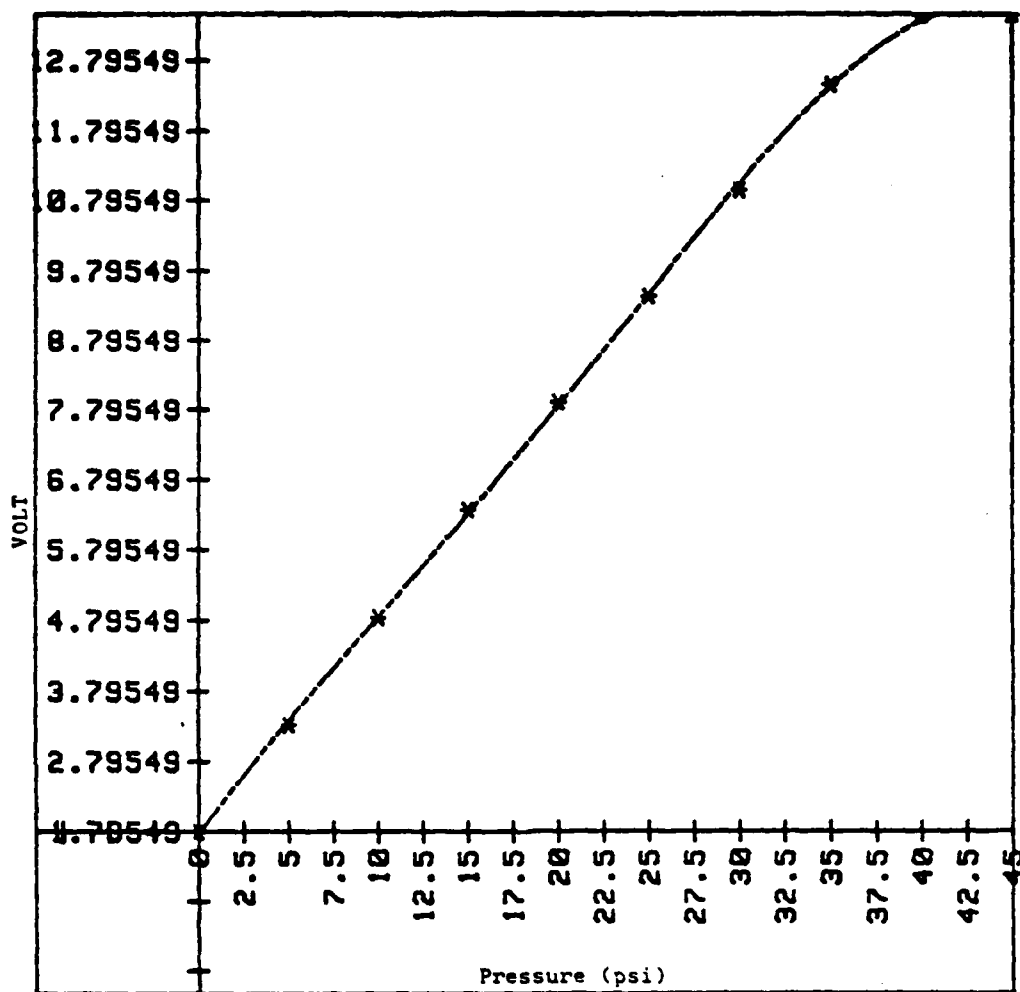
Calibration Plot of Pressure Transducer F



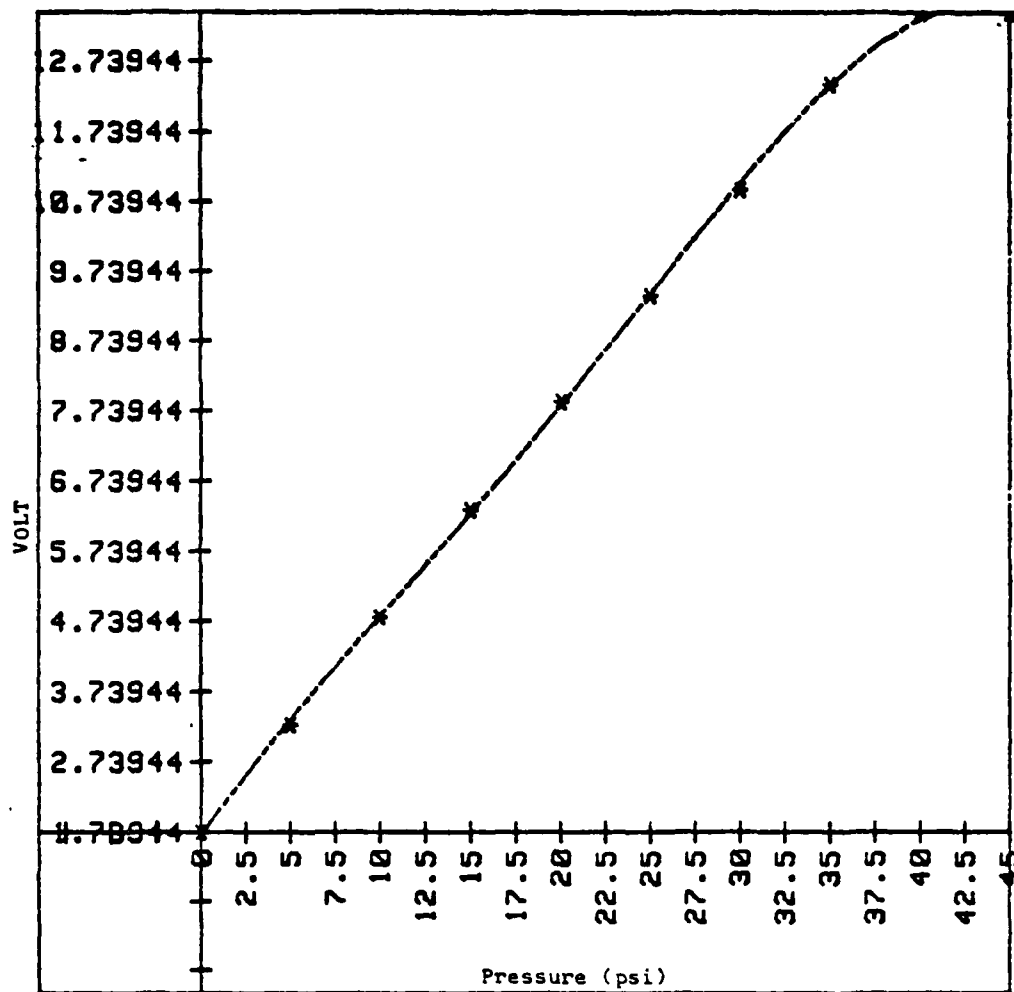
Calibration Plot of Pressure Transducer G



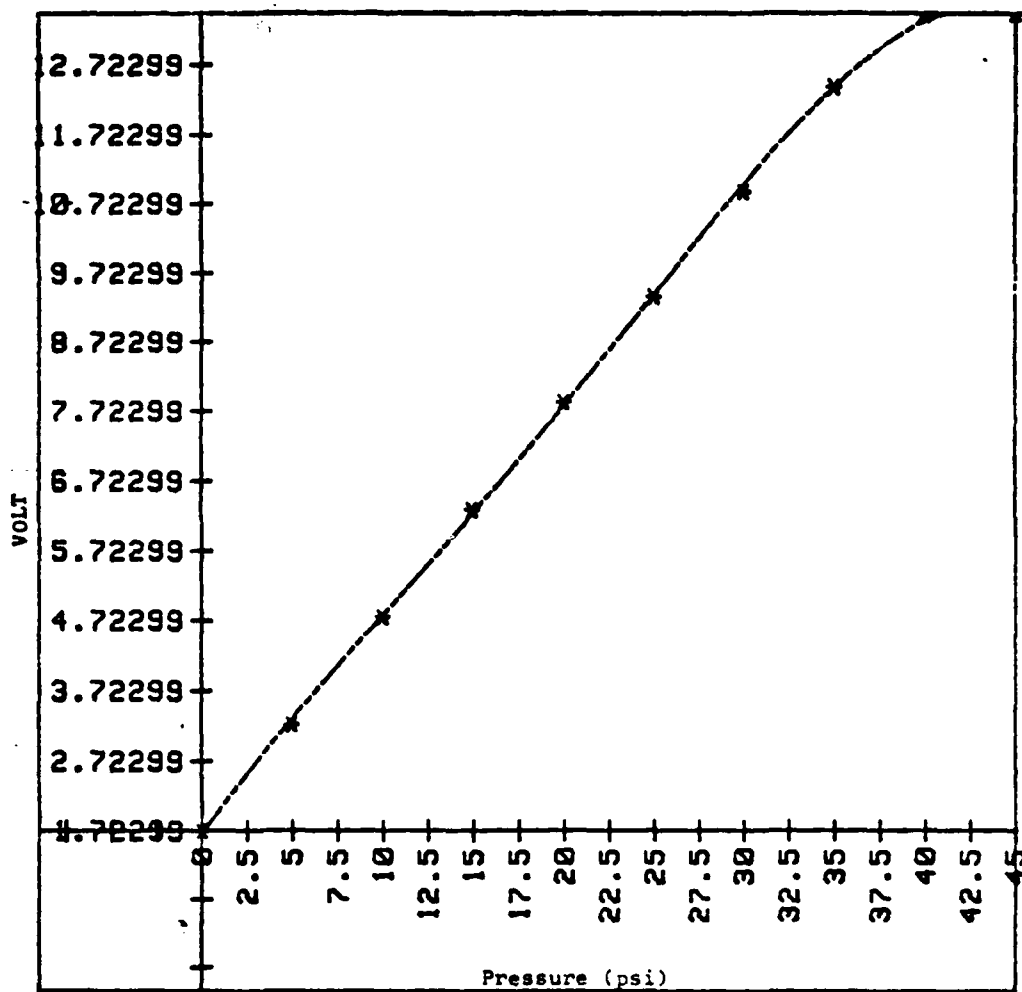
Calibration Plot of Pressure Transducer H



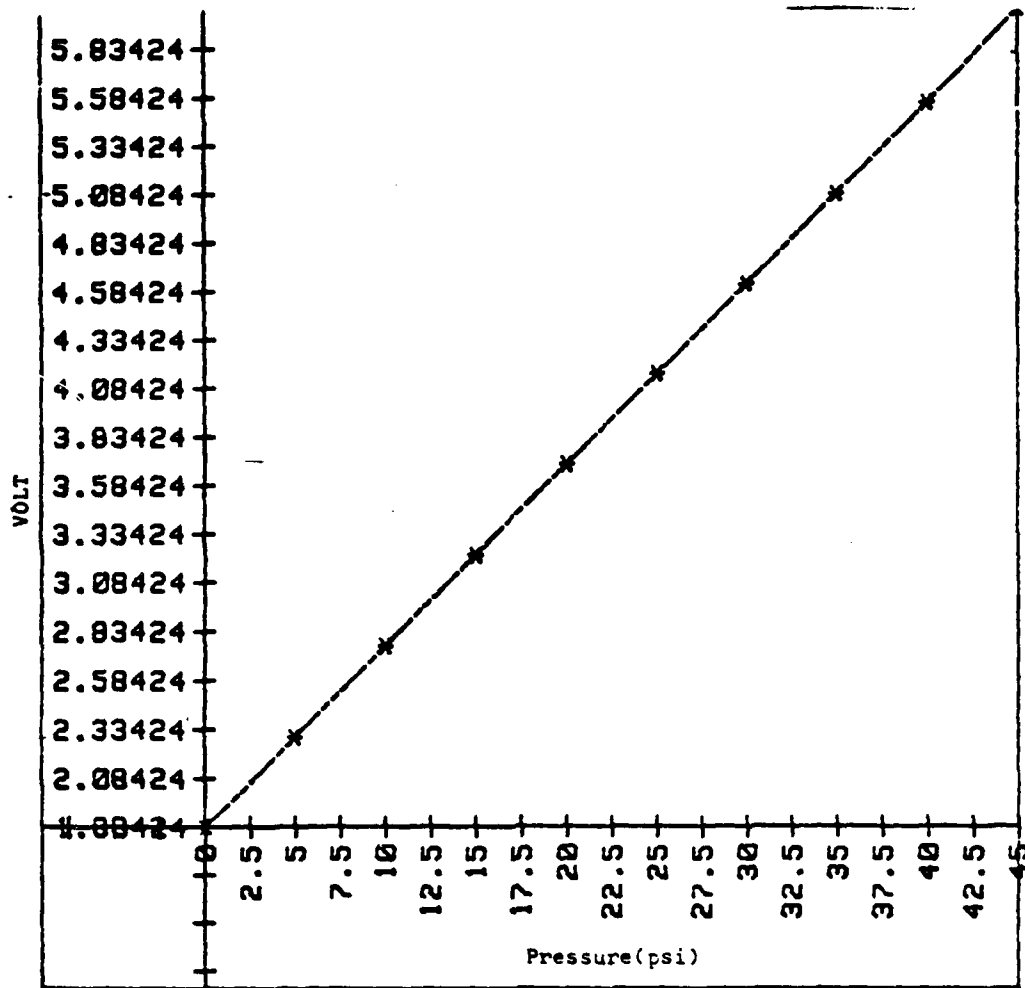
Calibration Plot of Pressure Transducer I



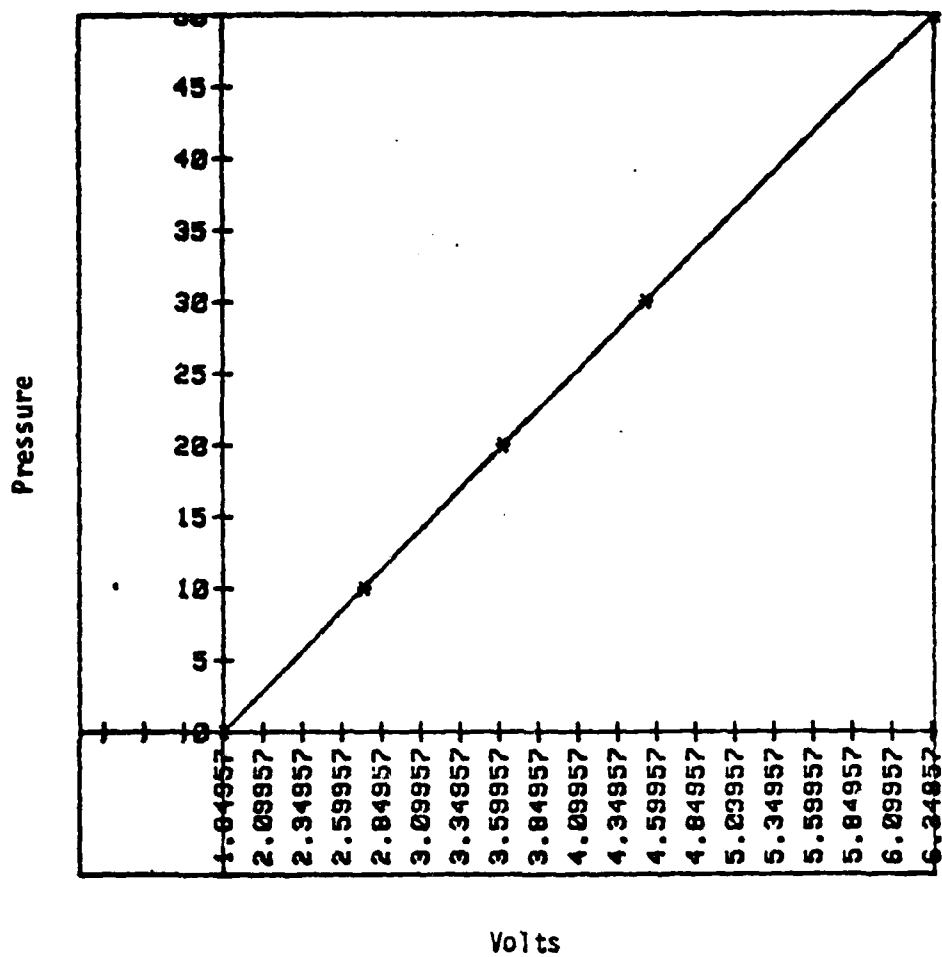
Calibration Plot of Pressure Transducer J



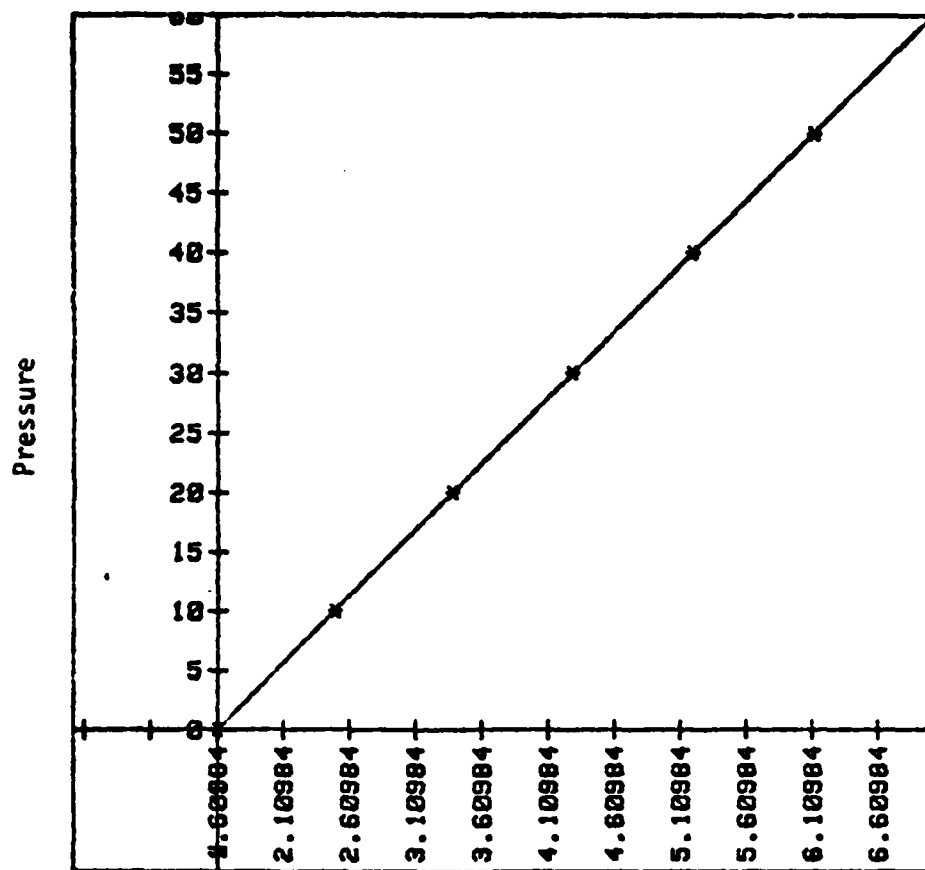
Calibration Plot of Pressure Transducer K



Calibration Plot of Pressure Transducer L



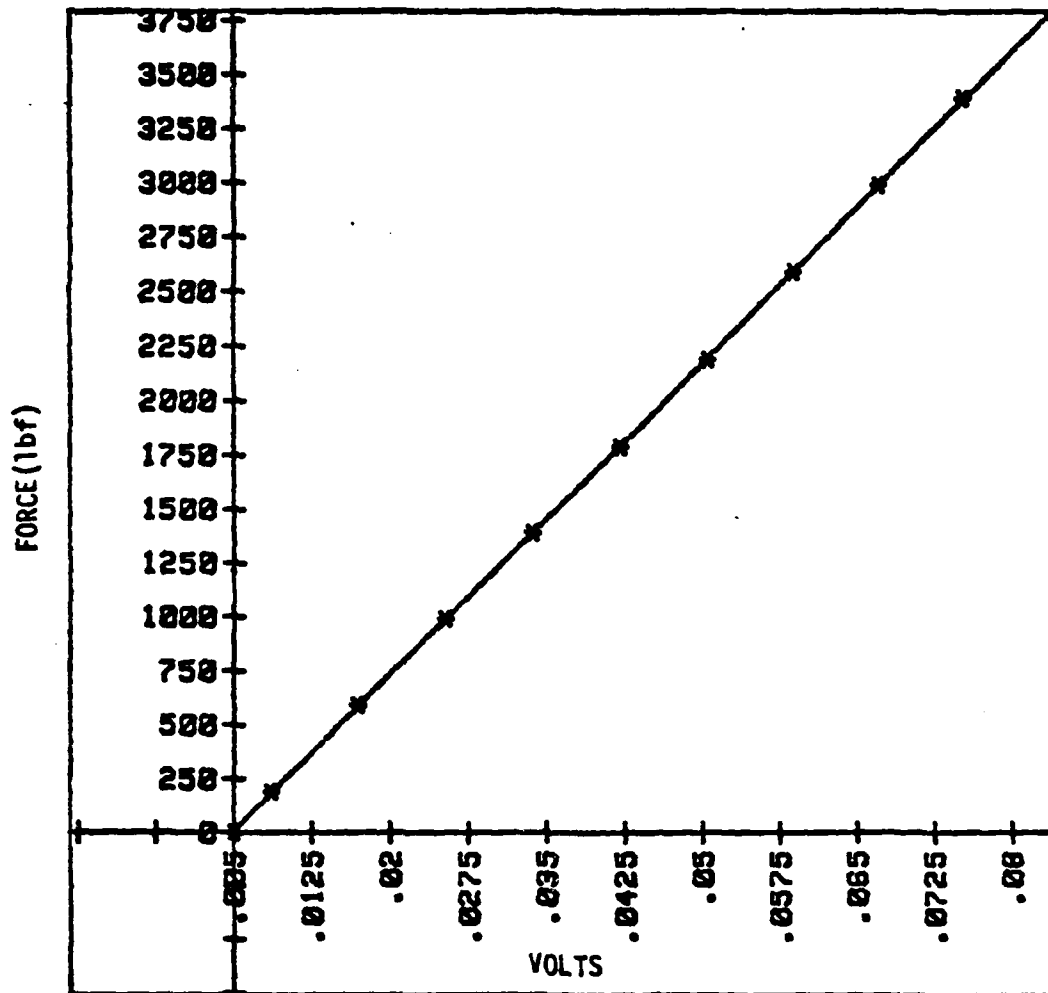
Calibration plot of pressure Transducer 0



Volts

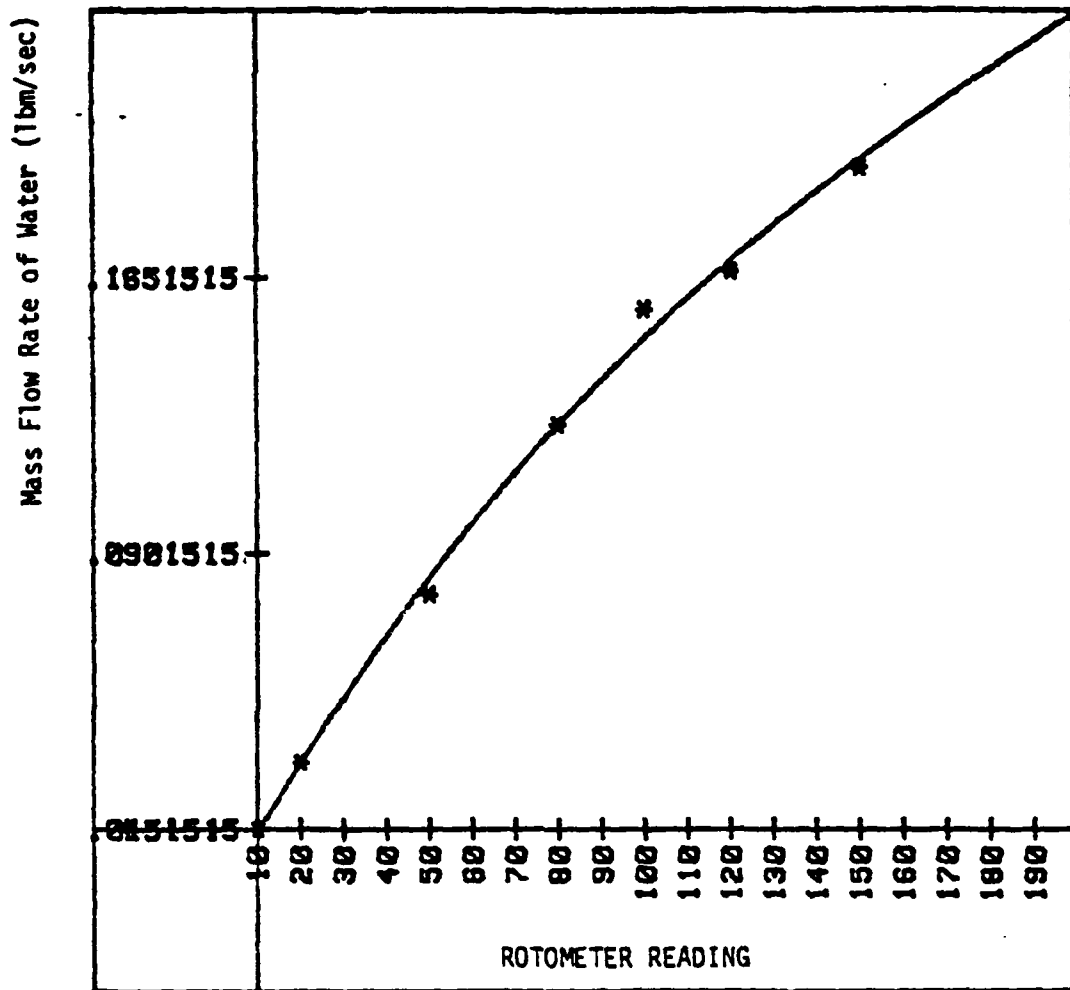
Calibration Plot of Pressure Transducer P

APPENDIX J: CALIBRATION PLOT FOR FORCE-BLOCK



Calibration Plot For Force-Block

APPENDIX K: CALIBRATION PLOT FOR ROTOMETER



Calibration Plot for Rotometer

APPENDIX L: DATA ACQUISITION AND ANALYSIS PROGRAM

```

3  PRINT "INPUT ROTOMETER READING"
4  INPUT Rr
5  PRINT Rr
7  DIM A(14),B(14),C(14),Dist(14),D(14)
8  DIM Press(14)
10 DIM X(18,100),Sum(18),Mean(18),Sum2(18)
30  N=10
40  FOR I=1 TO N
50      OUTPUT 709;"AI1VT1"
60      ENTER 709:X(1,I)
70      OUTPUT 709;"AI2VT1"
80      ENTER 709:X(2,I)
90      OUTPUT 709;"AI3VT1"
100     ENTER 709:X(3,I)
110     OUTPUT 709;"AI4VT1"
120     ENTER 709:X(4,I)
130     OUTPUT 709;"AI5VT1"
140     ENTER 709:X(5,I)
150     OUTPUT 709;"AI6VT1"
160     ENTER 709:X(6,I)
170     OUTPUT 709;"AI7VT1"
180     ENTER 709:X(7,I)
190     OUTPUT 709;"AI8VT1"
200     ENTER 709:X(8,I)
210     OUTPUT 709;"AI9VT1"
220     ENTER 709:X(9,I)
230     OUTPUT 709;"AI10VT1"
240     ENTER 709:X(10,I)
250     OUTPUT 709;"AI11VT1"
260     ENTER 709:X(11,I)
261     OUTPUT 709;"AI12VT1"
262     ENTER 709:X(12,I)
263     OUTPUT 709;"AI13VT1"
264     ENTER 709:X(13,I)
265     OUTPUT 709;"AI14VT1"
266     ENTER 709:X(14,I)
267     OUTPUT 709;"AI15VT1"
268     ENTER 709:X(15,I)
270  NEXT I
440  FOR J=1 TO 16
450      Sum(J)=0
460      FOR I=1 TO N
470          Sum(J)=Sum(J)+X(J,I)
480      NEXT I
490      Mean(J)=Sum(J)/N
510  NEXT J

```



```

520 FOR I=1 TO 14
530 READ A(I),B(I),C(I),D(I),Dist(I)
540 DATA -6.049177,3.92598,-.1736131,.0112105,.5
550 DATA -7.059265,4.233834,-.2292065,.01415403,1.5
560 DATA -6.4905288,3.93412037,-.160249802,.009448598,2.5
570 DATA -7.05653853,4.69207328,-.341175732,.0189756532,3.5
580 DATA -7.00950749,4.15163753,-.207173977,.0123697788,4.5
590 DATA -6.38855694,3.93348997,-.160546739,.0095126941,5.5
600 DATA -6.58938462,3.95696947,-.164270383,.0096133938,6.5
610 DATA -6.587731,3.90750953,-.158062598,.0094269372,7.5
620 DATA -6.59491204,3.94889685,-.162699726,.009565139,8.5
630 DATA -6.35328646,3.92659595,-.165085722,.0099773509,9.5
640 DATA -6.20324295,3.84628939,-.146968664,.0088046866,10.5
650 DATA -20.2065629,11.1571691,-.0859126,.004917379,0
660 DATA -20.6682414,11.0677155,-.07562378,.011185984,0
670 DATA -18.8267355,11.9281137,-.17715903,.011005888,0
680 Press(I)=A(I)+B(I)*Mean(I)+C(I)*Mean(I).2+D(I)*Mean(I).3
690 Press(I)=Press(I)+14.696
700 NEXT I

```

```

700 Fb=1/453.6*(-234.396482+48715.7753*Mean(15)-11261.538*Mean(15)^2+33899.3*M
ean(15)^3)
710 Mwater=-.0063268+.002097278*Rr-.00000658*Rr^2+.000000011*Rr^3
720 Fn=Fb*22.5/22.8
730 Dw=.920/12
740 Dd=3/12
750 E=1/(1-Dw/Dd)
760 Dp=Mean(14)-Mean(13)
770 Cc=.62
771 K=Cc*E
772 Aa=3.1416*Dw^2/4
774 Y=1-(-.41+.35*(Dw/Dd)^4*(Dp/(1.4*Mean(14))))
780 Gc=32.2
790 Den=.076297*Mean(12)/14.696
800 Mair=K*Aa*Y*SQR(2*Gc*Den*Dp*144)
810 Mtotal=Mair+Mwater
820 Vexit=Fn/(Mtotal)*32.2
830 PRINT
840 PRINT "PRESSURE AS A FUNCTION OF DISTANCE"
850 PRINT " "
860 PRINT "DISTANCE(X)", "PRESSURE"
870 PRINT " "
880 FOR I=1 TO 14
890 PRINT Dist(I), Press(I)
900 NEXT I
910 PRINT "MASS FLOW RATE OF WATER"
920 PRINT "MASS FLOW RATE OF AIR"
930 PRINT "TOTAL MASS FLOW RATE"
940 PRINT "THRUST"
950 PRINT "MIXTURE RATIO"
960 PRINT "EXIT VELOCITY"
970 PRINT "INLET PRESSURE"
980 END
";Mwater;"LBM/SEC"
";Mair;"LBM/SEC"
";Mtotal;"LBM/SEC"
";Fn;"LBF"
";Mwater/Mair
";Vexit;"FT/SEC"
";Press(12);"PSI"

```

INPUT ROTOMETER READING
0

PRESSURE AS A FUNCTION OF DISTANCE

DISTANCE(X)	PRESSURE
.5	14.5598137417
1.5	14.4607948757
2.5	14.4788575371
3.5	14.4188520649
4.5	14.4711792506
5.5	14.5553698274
6.5	14.4532490549
7.5	14.4461409662
8.5	14.5336111051
9.5	14.5262956477
10.5	14.5245848183
0	14.1098474897
0	11.8981971096
0	17.6235437008
MASS FLOW RATE OF WATER	= .0063268 LBM/SEC
MASS FLOW RATE OF AIR	= .0110711550501 LBM/SEC
TOTAL MASS FLOW RATE	= .00474435505015 LBM/SEC
THRUST	= 1.76920486692 LBF
MIXTURE RATIO	= .571467021403
EXIT VELOCITY	= 12007.6166544 FT/SEC
INLET PRESSURE	= 14.11058474897 PSI

APPENDIX M: DUAL-PHASE TWO-COMPONENT COMPUTER PROGRAM

```

SJOB
C.....
C REAL TWO-PHASE TWO-COMPONENT MODEL FLOW
C SHORTED FOR USE AT NAVAL POSTGRADUATE SCHOOL
C PROF. J. SLADKY AND LT. T. C. MULLIS
C.....
C COMMON ZZZZAY
C DIMENSION ZZZZAY(11200)
C THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTOP
C IT MUST BE THE FIRST COMMON BLOCK
COMMON A, ALPHS, AM, ALPH, ALPHAB, ALPHA, ALPHNO
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100 101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116 117 118 119 120 121 122 123 124 125 126 127 128 129 130 131 132 133 134 135 136 137 138 139 140 141 142 143 144 145 146 147 148 149 150 151 152 153 154 155 156 157 158 159 160 161 162 163 164 165 166 167 168 169 170 171 172 173 174 175 176 177 178 179 180 181 182 183 184 185 186 187 188 189 190 191 192 193 194 195 196 197 198 199 200 201 202 203 204 205 206 207 208 209 210 211 212 213 214 215 216 217 218 219 220 221 222 223 224 225 226 227 228 229 230 231 232 233 234 235 236 237 238 239 240 241 242 243 244 245 246 247 248 249 250 251 252 253 254 255 256 257 258 259 260 261 262 263 264 265 266 267 268 269 270 271 272 273 274 275 276 277 278 279 280 281 282 283 284 285 286 287 288 289 290 291 292 293 294 295 296 297 298 299 300 301 302 303 304 305 306 307 308 309 310 311 312 313 314 315 316 317 318 319 320 321 322 323 324 325 326 327 328 329 330 331 332 333 334 335 336 337 338 339 340 341 342 343 344 345 346 347 348 349 350 351 352 353 354 355 356 357 358 359 360 361 362 363 364 365 366 367 368 369 370 371 372 373 374 375 376 377 378 379 380 381 382 383 384 385 386 387 388 389 390 391 392 393 394 395 396 397 398 399 400 401 402 403 404 405 406 407 408 409 410 411 412 413 414 415 416 417 418 419 420 421 422 423 424 425 426 427 428 429 430 431 432 433 434 435 436 437 438 439 440 441 442 443 444 445 446 447 448 449 450 451 452 453 454 455 456 457 458 459 460 461 462 463 464 465 466 467 468 469 470 471 472 473 474 475 476 477 478 479 480 481 482 483 484 485 486 487 488 489 490 491 492 493 494 495 496 497 498 499 500 501 502 503 504 505 506 507 508 509 510 511 512 513 514 515 516 517 518 519 520 521 522 523 524 525 526 527 528 529 530 531 532 533 534 535 536 537 538 539 540 541 542 543 544 545 546 547 548 549 550 551 552 553 554 555 556 557 558 559 560 561 562 563 564 565 566 567 568 569 570 571 572 573 574 575 576 577 578 579 580 581 582 583 584 585 586 587 588 589 590 591 592 593 594 595 596 597 598 599 600 601 602 603 604 605 606 607 608 609 610 611 612 613 614 615 616 617 618 619 620 621 622 623 624 625 626 627 628 629 630 631 632 633 634 635 636 637 638 639 640 641 642 643 644 645 646 647 648 649 650 651 652 653 654 655 656 657 658 659 660 661 662 663 664 665 666 667 668 669 670 671 672 673 674 675 676 677 678 679 680 681 682 683 684 685 686 687 688 689 690 691 692 693 694 695 696 697 698 699 700 701 702 703 704 705 706 707 708 709 710 711 712 713 714 715 716 717 718 719 720 721 722 723 724 725 726 727 728 729 730 731 732 733 734 735 736 737 738 739 740 741 742 743 744 745 746 747 748 749 750 751 752 753 754 755 756 757 758 759 760 761 762 763 764 765 766 767 768 769 770 771 772 773 774 775 776 777 778 779 780 781 782 783 784 785 786 787 788 789 790 791 792 793 794 795 796 797 798 799 800 801 802 803 804 805 806 807 808 809 810 811 812 813 814 815 816 817 818 819 820 821 822 823 824 825 826 827 828 829 830 831 832 833 834 835 836 837 838 839 840 841 842 843 844 845 846 847 848 849 850 851 852 853 854 855 856 857 858 859 860 861 862 863 864 865 866 867 868 869 870 871 872 873 874 875 876 877 878 879 880 881 882 883 884 885 886 887 888 889 890 891 892 893 894 895 896 897 898 899 900 901 902 903 904 905 906 907 908 909 910 911 912 913 914 915 916 917 918 919 920 921 922 923 924 925 926 927 928 929 930 931 932 933 934 935 936 937 938 939 940 941 942 943 944 945 946 947 948 949 950 951 952 953 954 955 956 957 958 959 960 961 962 963 964 965 966 967 968 969 970 971 972 973 974 975 976 977 978 979 980 981 982 983 984 985 986 987 988 989 990 991 992 993 994 995 996 997 998 999 1000 1001 1002 1003 1004 1005 1006 1007 1008 1009 1010 1011 1012 1013 1014 1015 1016 1017 1018 1019 1020 1021 1022 1023 1024 1025 1026 1027 1028 1029 1030 1031 1032 1033 1034 1035 1036 1037 1038 1039 1040 1041 1042 1043 1044 1045 1046 1047 1048 1049 1050 1051 1052 1053 1054 1055 1056 1057 1058 1059 1060 1061 1062 1063 1064 1065 1066 1067 1068 1069 1070 1071 1072 1073 1074 1075 1076 1077 1078 1079 1080 1081 1082 1083 1084 1085 1086 1087 1088 1089 1090 1091 1092 1093 1094 1095 1096 1097 1098 1099 1100 1101 1102 1103 1104 1105 1106 1107 1108 1109 1110 1111 1112 1113 1114 1115 1116 1117 1118 1119 1120

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2 DIMENSION PL(10) , Y0 , YOS , ZZZ , NSTMT
DIMENSION NGRAY(75)
DIMENSION DATE(5), CACT(3), CID(8), KP(2,75), PL(1,13)
1 C(16), AD(2,75), KXA(4), DGRAY(100), DSRAY(100), TGRAY(100), TGRAY
2 (100), DGRAY(75), HT(14), DINT(8)
EQUIVALENCE (NDBAT(1), DGRAY(1))
XX=-1.0
XMIN = 1.0E-6
IXMIN = 0
YMIN = 0
DINT(1) = .5
DINT(2) = .1
DINT(3) = .01
DINT(4) = .001
DINT(5) = .0001
DINT(6) = .00001
DINT(7) = .000001
DINT(8) = .0000001
CALL ADATA
C INITIALIZE TABLE TAPP, BRING IN PERMANENT TABLES (ONE-DIMENSIONAL),
C FILL ARRAY INDICATING TEMPERATURES INCLUDED IN TWO
C DIMENSIONAL TABLES, INITIALIZE PAGE COUNT.
5 LOGCT=1
CALL TABLE
CALL INTERP (VAR,0,2,P)
C ACTIVATE THE FOLLOWING CARD TO GET A PRINTOUT OF THE PROPERTIES
CALL INTERP (VAR,20,7,C)
NSTMT=1999
100 CONTINUE
CALL SECT1
IF (NSTMT-5) 999,5,101
101 CONTINUE
IF (NSTMT-99) 999,99,200
999 CALL SUMF
200 CONTINUE
CALL SECT2
IF (NSTMT-63) 999,300,201
201 IF (NSTMT-52) 999,300,202
202 IF (NSTMT-31) 999,400,203
203 IF (NSTMT-15) 999,700,999
300 CONTINUE
CALL SECT3
IF (NSTMT-32) 999,701,101
101 IF (NSTMT-11) 999,700,102
102 IF (NSTMT-145) 999,700,104
104 IF (NSTMT-161) 999,800,105
305 IF (NSTMT-1142) 999,200,999
400 CONTINUE
CALL SECT4
IF (NSTMT-10) 999,200,401
401 IF (NSTMT-11) 999,700,402
402 IF (NSTMT-123) 999,600,403
403 IF (NSTMT-161) 999,200,999
500 CONTINUE
CALL SECT5
IF (NSTMT-10) 999,200,501
501 IF (NSTMT-11) 999,700,502
502 IF (NSTMT-239) 999,600,999
600 CONTINUE
CALL SECT6
IF (NSTMT-103) 999,500,601
601 IF (NSTMT-113) 999,700,602
602 IF (NSTMT-62) 999,800,602
602 IF (NSTMT-1490) 999,500,999
700 CONTINUE
CALL THERCAT
IF (NSTMT-497) 999,800,999
800 CONTINUE

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FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

Year	1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	2100
1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	2100	

[illegible][illegible]

744, A2)
8077 FORMAT(1H0,10X,19HISOTHERMAL FLOW TO ,1PE15.4 , 6H PSTA.)
END

SUBROUTINE SECT2

COMMON TZZZAY
DIMENSION TZZZAY(11210)
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTRP
IT MUST BE THE FIRST COMMON BLOCK

C	COMMON	A	ADAR	ALAN	ALPHAB	ALPHA	ALPHMB
1		ALPHS	AM	AP	AS	AT	BA
2		BB	BETAB	BETAMB	BETA	BETAS	BB
3		C	CI	CAH1	CAHNB	CAH	CALMB
4	CBG1	CAL	CASE	CBGMB	CBG	CBG	CDL
5	COMMON	CHNS	CHP	CD	CPIN	CPINS	CPIN
6		CFMS	CG43	CG4	CGMS	CGLS	CL
7		CLMS	CSAVE	D	DE	DE	DATE
8		DD	DELSI	DELI	DELS	DELT	DELS
9		DELOS	DELSI	DELSO	DELSN	DELSB	DELS2
1	COMMON	DETAGB	DETAG	DETAGB	DETAG	DETAG	DETAG
2		DELSIS	DELSOS	DO	DEI	DEI	DEI
3		DSI	DRO	DR	DSI	DSAVE	DSI
4		DSMAX	DSMIN	DSRAY1	DSRAY	DSRAYS1	DSRAYS2
5	COMMON	DSIES	DS	DSX1	DSX2	DSX1	DSX2
6		DTG	DTU	DTAG	DTL1	DTLB	DTG
7		DTQ	DVBSQ	DVBSQ	DVLSO	DA	DTI
8		EPBG	EMGB	EM	EMGB	EMGB	EMGB
9		EML	EMLS	EMT	EMT	EMT	EMT
1	COMMON	EM	EM	EM	EM	EM	EM
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DIMENSION NDRAY(75)
1 DIMENSION DATP(5), CASE(3), QID(9), TP(2,75), PL1(10)
2 C(6), A2(2,75), KXA(8), DSRAY(100), DSRAY1(100), TDRAY(100), TDRAY1(100)
EQUIVALENT(75), HZ(14), DINT(9)
IF(NS=1-10) 1,30,2
1 CALL DUMF
2 IF(NS=1-145) 1,145,3
3 IF(NS=1-161) 1,161,4
4 IF(NS=1-1102) 1,1142,5
5 CONTINUE

SET ITERATION COUNT ZERO
NNI=0

SET INITIAL CONDITIONS INDICATION
***** STATEMENT NO. 20 *****
20 IFP=1
  IFP=1
  ISL=0
  KAI=1
  NNI=0
  SO=0.1
  PHI=PHI0.017 3329
  AT=103300.3

INITIALIZE PRINTOUT COUNT
NMP=1

SETUP MIDPOINT AND ENDPOINT INDICATION.
INC = 1, NEXT ENTRY TO STATEMENT 32 COMPUTES MIDPOINT
QUANTIFIED.
INC = 1, NEXT ENTRY TO STATEMENT 32 COMPUTES ENDPOINT
QUANTIFIED.
IN=1

RESET THROAT INDICATOR
THRT = 0 - THROAT NOT REACHED
THRT = 1 - THROAT REACHED
THRT=0

RESET DS ITERATION COUNTER.
NDS=0
RESET OPTIMUM S ITERATION COUNTER.
NSO=0

BACKSTEP P FOR FIRST ENTRY TO 30
P = PD-DP1/2.3

SETUP INITIAL CONDITIONS
28 IS=TQ
  IL=TL0
  VG=VLO
  VL=VLO
  D=0
27 I=0.
  DTL=0.0
  DS=0.
  R=0.
  DTG=0.0
  AT=0.

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VGSQ=VG*VG
VLSQ=VL*VL
THQ=THQO
THI=THIO
DELTV=VG-TL
DELVL=VGSQ-VLSQ
DSI=0.
R=0.
DTMO=0.0
DTMI=0.0
***** STATEMENT NO. 10 *****
10 NSTAT=9999
IF (MPB) 611,610,630
THIS IS FIRST POINT - HALF STEP P BY INITIAL DP STEP-SIZE.
611 P=P+DP1/2.0
GO TO 632
THIS IS NOT FIRST POINT - HALF STEP PRESSURE BY STEP
SIZE DP.
630 P=P+DP/2.0
632 IF (P) 1130,1130,1131
STEPPING PRESSURE MADE IT NEGATIVE - PRINT DIAGNOSTIC, AND
DATA COMPUTED TO THIS POINT AND GO TO NEXT CASE.
1130 WRITE (6,930)
LL=1
GO TO 500
EXTRAPOLATE TG AND TL, PRINT DIAGNOSTICS IF THEY GO
NEGATIVE, AND PRINT DATA COMPUTED TO THIS POINT.
131 CONTINUE
TG=TO+DTG/2.0
IF (TG) 132,132,133
132 WRITE (6,931)
LL=2
GO TO 500
133 TL=TL+DTL/2.0
IF (TL) 134,134,135
134 WRITE (6,932)
LL=3
GO TO 500
INTERPOLATE FOR PBO FOR THIS TL
135 N=9
NN=1
CALL INTER(PBO,N,TL,0)
IF (N) 136,136,137
136 WRITE (6,913) NT(NN),TL,P
LL=1
GO TO 500
CHECK DENOMINATOR OF EQUATION FOR PARTIAL PRESSURE
OF COMPONENT A, PRINT DIAGNOSTIC IF SUFFICIENTLY
CLOSE TO ZERO RENDERING PA INFINITE.
137 IF (ABS(1.0-H*PBO)-.0001) 138,138,139
138 WRITE (6,938)
LL=1
GO TO 500
139 PA=(P-PBO)/(1.0-H*PBO)
COMPUTE PA AND PB. IF EITHER IS NEGATIVE, PRINT
DIAGNOSTIC AND GO TO NEXT CASE.

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0	0	1	4	9	16	25	36	49	64	81	100	121	144	169	196	225	256	289	324	361	400	441	484	529	576	625	676	729	784	841	900	961	1024	1089	1156	1225	1296	1369	1444	1521	1600	1681	1764	1849	1936	2025	2116	2209	2304	2401	2500	2601	2704	2809	2916	3025	3136	3249	3364	3481	3600	3721	3844	3969	4096	4225	4356	4489	4624	4761	4900	5041	5184	5329	5476	5625	5776	5929	6084	6241	6400	6561	6724	6889	7056	7225	7396	7569	7744	7921	8100	8281	8464	8649	8836	9025	9216	9409	9604	9801	10000

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CC TABLE LIMITS EXCEEDED - WRITE DIAGNOSTIC.
160 LL=8
161 WRITE (6,2136) HT(NN),TL,P
162 N=11
GO TO 500

CC INTERPOLATE IN TABLE FOR DENSITY OF LIQUID B AT THIS
TEMPERATURE.
162 NN=11
CALL INTER (ROBL,N,TL,P)
IF (N) 161,161,163

CC COMPUTE DENSITY OF LIQUID MIXTURE
163 RHOL=1.0/((ALPHA/ROAL)+(1.0-ALPHA)/ROBL)

CC COMPUTE FLOW RATES OF LIQUID MIXTURE AND OF GAS MIXTURE.
ENG=PNT/(1.1+R)
ENC=P*ENG

CC COMPUTE RATIO OF GAS VOLUME FLOW TO LIQUID VOLUME FLOW.
RV=RHOL/(R*RHOG)

CC INTERPOLATE IN SPECIFIC HEAT TABLES FOR A AND B AT
THIS TEMPERATURE AND PRESSURE.
N=1
NN=1
CALL INTER (CAG,N,TG,PA)
IF (N) 165,165,170
165 GO TO 1142
170 N=2
NN=2
CALL INTER (CBG,N,TG,PB)
IF (N) 165,165,171
171 INTERPOLATE INTO LATENT HEAT OF VAPORIZATION TABLES
FOR A AND B AT THIS TEMPERATURE.
N=7
NN=7
CALL INTER (HLA,N,TL,P)
IF (N) 161,161,172
172 N=8
NN=8
CALL INTER (HLB,N,TL,P)
IF (N) 161,161,31
31 IF (N-1) 12,40,32
32 IF (N-1) 60,50,60
40 NST=40
50 NST=50
60 NST=60
RETURN
500 *DIAG=1
CALL JUDGEUT (NN,N,LIQCT,NG20)
NST=110
RETURN
910 FORMAT (1H0,7X,11HNONPOSITIVE P)
911 FORMAT (1H0,7X,14HNONPOSITIVE TG)
912 FORMAT (1H0,7X,14HNONPOSITIVE TL)
913 FORMAT (1H0,7X,11HINFINITE PA)
9136 FORMAT (1H0,7X,17HOUTSIDE RANGE OF ,A6,7H TABLE,5X,

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2      VIS      , YO      , YOS      , ZZZ      , NSTTT      SEC3 0690
      DIMENSION PL(10) , QD(8) , KP(2,75) , PL1(10) ,
      DIMENSION DATE(3) , CASE(3) , QD(8) , KP(2,75) , PL1(10) ,
      DIMENSION C(16) , AP(2,75) , XTA(8) , DSRAT(100) , DSRAT(100) , THRAT(100) , TBRAT(100) ,
      1 1000) , ORAT(75) , HT(18) , DINT(8) ,
      EQUIVALENCE (NDRAT(1) , ORAT(1))
      IF (NSTTT-40) 1,40,2
      1 CALL DUFF
      2 IF (NSTTT-50) 1,50,1
      THIS IS FIRST POINT - STORE INITIAL PROPERTIES FOR USE
      AT END OF FIRST INTERVAL.
      ***** STATEMENT NO. 40 *****
      40 NSTTT=4999
      HLA1=HLA
      HLB1=HLB
      CAG1=CAG
      CEG1=CEG
      COMPUTE INITIAL FLOW RATES OF COMPONENTS A AND B.
      EMAG=(EMG*WAG*PA)/(WG*P)
      EMB=(EMG*WBG*PB)/(WB*P)
      DR=1.0
      174 IF (V1) 175,175,178
      175 IF (V2) 176,176,178
      175 A=1000.000
      RA=A
      GO TO 41
      COMPUTE INITIAL AREA OF NOZZLE.
      176 A=144. *EMG*((1.0/(RHOG*VG)) + (R/(RHOL*VL)))
      RA=RV*VL/VG
      41 AT=A
      DT=0
      AP(1,1)=F
      AP(1,1)=A
      KAP=2
      COMPUTE SLIP VELOCITY, VS, MEAN FREE STREAM VELOCITY, VB, AND
      SLIP FRACTION, S.
      42 VS=VG-VL
      VB=(VG+K*VL)/(1.0+K)
      VBSO=V*VB
      S=VS/VB
      IF (KGT0) 180,179,180
      NOZZLE IS CIRCULAR - COMPUTE DISTANCE FROM NOZZLE AXIS TO
      WALL OF NOZZLE, YI.
      179 YI=SQRT(A/3.1416)
      YI=1.0
      YI=0.0
      GO TO 43
      NOZZLE IS ANNULAR - COMPUTE RADII AND DISTANCES TO WALL
      OF ANNULAR NOZZLE, RI,RO,YI,YO.
      180 IST=(RAIC**2)-((A*COS(PHI))/(2.0*3.1416))
      IF (IST) 181,182,182
      181 *PI=180.0,181.0
      LL=11
      GO TO 500
      182 RI=SQRT(IST)
      RO=(RATIO**2) + ((A*COS(PHI))/(2.0*3.1416))
      RO=SQRT(RO)

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      YI=(XAXO-HI)/COS(PHI)
      YO=(H0-PAXO)/COS(PHI)
C   WRITE LINES 1,2,3 OF OUTPUT AND GO TO 33.
C   43 IF (LCT1-42) 644,643,641
C   643 LCT1=0
C       WRITE (6,9543) LPGCT
C       LPGCT=LPGCT+1
C   644 WRITE (7,1331) X,P,R,V3,A,TG,TL,VG,VL
C       WRITE (7,1332) VS,S,V,AV,AA,ALPHA,BETA,ENG,ENL
C       WRITE (7,1333) RHCG,RHOL,WAG,WBG,W,PA,PB,HLA,HLB
C       LCT1=LCT1+9
C       GO TO 30
C   MIDPOINT OF INTERVAL - COMPUTE MID-POINT PARAMETERS.
C   RESET MIDEQINT/ENDPOINT INDICATOR SO NEXT ENTRY TO 32
C   WILL BE AN ENDPOINT.
C   ***** STATEMENT NO. 50 *****
C   50 INT=0
C       NST=9999
C   INTERPOLATE IN VISCOSITY (GAS) TABLES FOR A AND B AT
C   THIS GAS TEMPERATURE.
C       N=16
C       NN=16
C       CALL INTRP(VIAG,N,TG,PA)
C       IF (N) 205,205,206
C   205 LL=12
C       GO TO 1142
C   206 N=17
C       NN=17
C       CALL INTRP(VIBG,N,TG,PB)
C       IF (N) 207,207,208
C   207 LL=12
C       GO TO 145
C   INTERPOLATE IN SPECIFIC HEAT TABLES FOR A AND B AT
C   THIS LIQUID TEMPERATURE.
C   208 N=5
C       NN=5
C       CALL INTRP(CAL,N,TL,P)
C       LL=12
C       IF (N) 161,161,209
C   209 NN=6
C       N=6
C       CALL INTRP(CBL,N,TL,P)
C       LL=12
C       IF (N) 161,161,210
C   COMPUTE MEAN SPECIFIC HEATS OF LIQUID, CLN, AND OF
C   GAS, CGN.
C   210 CLN=ALPHA*CAL+(1.0-ALPHA)*CBL
C       CGN=(1.0-BETA)*CGA+(BETA)*CGG
C       PHIA=1/(1.0+((VIAG/VIBG)**.5)*((WBG/WAG)**.25)**2)/
C       1.224*{(1.0+AV/AG)**.5)}
C       PHIB=1/(1.0+((VIBG/VIAG)**.5)*((WAG/WBG)**.25)**2)/
C       1.224*{(1.0+VB/BG)**.5)}
C   COMPUTE MEAN VISCOSITY OF A AND B GAS.
C       VIGH=(VIAG/(1.0+(BETA/(1.0-BETA))*((WAG/WBG)*PHIA)))+
C       1. VIBG/(1.0+(((1.0-BETA)/BETA)*((WBG/WAG)*PHIB)))
C   INTERPOLATE IN VISCOSITY (LIQUID) TABLES FOR A AND B AT
C   THIS TEMPERATURE.

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1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	8												

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DSMAX = 1.0 * 2 - 1.0E-06
GO TO 70
C
SPECIFIED X(P) IS TO BE USED.
52 P10=P*DP/2.0
P11=P*DP/2.0
PINT=P10
KL=1
GO TO 1100
1152 X10=XINT
PINT=P11
KL=2
GO TO 1100
1153 X11=XINT
C
GET DIFFERENCE OF INTERPOLATED X VALUES FROM
X(P) TABLE
C
DX=X10-X11
S1=(0.75*RHOG*(1.0+R)*DX)/(D*RHOL)
S2=(4*H33.04*(1.0+R)*DP)/RHOL
S3=(1.0+R)/2.0
S4=1.0*DS/(1.0+R)
GO TO 10070
C
INTERPOLATE INTO X(P) TABLE FOR TWO PRESSURE VALUES
AT EITHER SIDE OF CURRENT INTERVAL.
C
1100 CONTINUE
THIS CHANGE TO ALLOW FOR MONOTONIC INCREASING OR DECREASING TABLE
TERZ=PINT-XP(1,1)*1.0E-9
IF (TERZ*(XP(1,1)-XP(1,2))) 1106,1102,1105
1106 CONTINUE
DO 1103 KR=2,KXP1
TFR=XP(1,KR)-PINT
IF (TERZ*TFR) 1101,1104,1104
1101 TFR=-TFR
1103 CONTINUE
GO TO 1120
1102 CONTINUE
KR=2
R=1+R-1
4INT=XP(2,KR1)+TERZ/(XP(1,KR)-XP(1,KR1))*(XP(2,KR)-XP(2,KR1))
GO TO (1152,1153),KL
1105 CONTINUE
IF (TERZ<0.001) 1108,1120,1120
IF (TERZ>0.001) 1120,1120,1113
1110 TERZ=-1.00*TERZ
GO TO 1106
1120 *RINT (6,9105) P10,P11,KL,TERZ,TERY,PINT
LL=14
GO TO 500
30 NST=10
RETURN
1142 NST=1142
RETURN
145 NST=145
RETURN
161 NST=161
RETURN
70 NST=70
RETURN
500 NDIAG=1
CALL OUTPUT(NNN,LPGCT,JGEO)
NST=110
RETURN
C
FORMAT STATEMENTS FOR 3-LINE PERMANENT OUTPUT.
1001 FORMAT (1H0,13X,1HX,13X,1HP,13X,1HR,12X,2HVB,13X,1HA,12X,2HTG,12X,
12HTL,12X,2HVG,12X,2HVL,/,6X,1P4E14.4)

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COMMON VIALD , VIAL , VIBG , VIBLB , VIBL , VIGN , SEC4 0670
6 VIGPS , VILH , VILN , VILNS , VLM , VLD , SEC4 0680
7 VL , VLSO , VLS , VS , VSS , WATD , SEC4 0690
8 WAT , WALS , WAL , WGBB , WAG , WBS , SEC4 0700
9 WCL , WCB , WGS , WIS , WIS , WIS , SEC4 0710
COMMON WGN , WNGS , Y , X10 , Y11 , VCLK , SEC4 0720
1 X11 , XP , XS , YX , XA , YI , SEC4 0730
2 YS , YO , YOS , ZZZ , ZSTMT , YI , SEC4 0740
DIMENSION PL(10)
DIMENSION TFRAY(75)
DIMENSION DACE(5) , CAS*(3) , JID(4) , XP(2,75) , PL(10) ,
1 CIB(12) , XA(18) , DSRAY(100) , TFRAY(100) , TFRAY(100) , TFRAY(100) , SEC4 0750
21(100) , DFRAY(75) , HT(11) , JINT(3)
EQUIVALENCE (NDRAY(1) , DFRAY(1))
IF (NDRAY-NH) 1,60,2
1 CALL DDFP
2 IF (NDRAY-NH) 1,63,1
C
C ***** STATEMENT NO. 60 *****
C
60 INT=1
NSTRT=9999
VB=ABS(SQRT(VBSQ))
C
C COMPUTE NEW SLIP FRACTION, S, AND NEW DISTANCE, X.
C
IF (NPRO-1) 4001,4000,4001
4000 S=S
GO TO 8002
4001 S=0.05
4002 X=X+DX
C
C COMPUTE CHANGES IN SQUARE OF GAS AND LIQUID VELOCITIES.
C
DVGSQ=((1.0*(R*S)/(1.0*R1))*2)*VBSQ-VGSQ
DVLSQ=((1.0-(R*S)/(1.0*R1))*2)*VBSQ-VLSQ
C
C COMPUTE CHANGES IN GAS AND LIQUID FLOW RATES.
C
DEMG=(DEMG*AG*PA)/(AG*P)-CMAG
DEPLG=(DEMG*AG*PB)/(AG*P)-CMPL
C
C COMPUTE CHANGES IN GAS AND LIQUID TEMPERATURES.
C
DTL=(1.0/CLM)*(((CM*DELT*HKL)/(500.0*DEHLM*VLM))
1 -(HLM*DEMG/DELM)-(HLM*DEMG/DELM))
DELT=DTL-TL
DTG=(-1.0/CEM)*((10785/50072.9)*R*
1 (CLM*DTL*DP/(5.4017*RHLM)*DVLSQ/50072.9)+
2 ((DEMG*DEMG)*DELT)/(50072.9)*R*
3 ((HLM*CG1*DELT)*DEMG*(HLM*CBG1*DELT)*
4 DEMG/DEMG))
IF ((1.0*(R*CLM/CEM))*((CM*DELT*HKL)/(500.0*DEHLM*VLM))
1 -(HLM*DEMG/DELM)-(HLM*DEMG/DELM))
8076 DTG = 0
IF (P-1) 8075,8076,8076
8074 CONTINUE
THIS LAST TEST ADDED TO DELETE NEXT TEST AT END OF NOZZLE
C
C CHANGE TO SAVE THE LAST P FOR PRINTOUT IN SECT1.
C
8075 CONTINUE
C
C COMPUTE NEW TEMPERATURES FOR LIQUID AND GAS.
C
C IF EITHER IS NEGATIVE PRINT DIAGNOSTIC AND EXIT
C
TO NEXT CASE.
VGSQ=VGSQ+DVGSQ
IF (VGSQ) 230,230,231
230 WRITE (6,9230)
LL=15
SEC4 1310
SEC4 1320
SEC4 1330

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231 GO TO 500
    VLSQ=VLSQ+DVLSQ
    IF (VLSQ) 232,232,233
232 WRITE (6,9232)
    LL=15
    GO TO 500
C
C      COMPUTE NEW VELOCITIES FOR LIQUID AND GAS.
C
233 VG=ABS (SCRT (VGSQ))
    VL=ABS (SCRT (VLSQ))
C
C      COMPUTE RATIO OF GAS FLOW AREA TO LIQUID FLOW AREA.
C
    RA=RV*(VL/VG)
C
C      COMPUTE NEW LIQUID AND GAS FLOW RATES.
C
    RHAG=RHAG+DEHAG
    RHOG=RHOG+DERHOG
    DELV2=VLSQ-VLSQ
C
C      SET UP INITIAL QUANTITIES FOR NEXT END-POINT.
C
    DTLL=DTL
    DTGL=DTG
    HLAG=HLA
    HLB1=HLB
    DELT=TL-TL
    CAGI=CAG
    CGLI=CGL
    VS=S*VR
C
C      INTERPOLATE INTO SURFACE TENSION TABLE AT THIS LIQUID
C      TEMPERATURE.
C
    N=18
    NN=14
    CALL INTERP(SIG,N,TL,2)
    LL=15
    IF (N) 16,16,240
C
C      IF DROPLET BREAKUP OPTION SPECIFIED, COMPUTE DROPLET
C      DIAMETER.
240 IF (NEU-1) 241,238,243
238 DZ=(.3174*SI)/((RHOG*VS**2))
    IF (D-2) 243,243,249
239 D=DZ
C
C      COMPUTE CHANGE IN NOZZLE CROSS-SECTIONAL AREA SINCE
C      LAST END-POINT.
241 DA=144.0*ENG*(1.0/(RHOG*VG)+R/(PHOL*VL))-A
C
C      COMPUTE MEAN AREA BETWEEN THIS AND PREVIOUS ENDPOINT.
C
    AM=A+DA/2.0
C
C      COMPUTE PRESENT AREA
C
    A=A+DA
    IF (NEU-1) 67,68,67
C
C      COMPUTE RADII AND DISTANCE FROM AXIS TO WALL OF
C      CIRCULAR NOZZLE.
C      ***** STATEMENT NO. 67 *****
67 DYO=SQRT (A/.1416)-YO
    RCH=YO+DYO/2.0
    DRO=DYO
    YO=YO+DYO
    RO=YO

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      MIN=7.0
      DELT=0.0
      DELSI=0.0
      FDELIN=0.0
      CFIM=0.0
      TWIN=0.0
      GO TO 64

UUUU  COMPUTE GEOMETRY OF ANNULAR NOZZLE AND BOUNDARY-LAYER
      GROWTH AND SHAP FOR INNER WALL.

      68 MAX=PAIC-X* SIN (PHI)
      IF (MAX**2-(A*COS (PHI))/(2.0*3.1416)) 241,242,242
241  WRITE (6,2241)
      LL=17
      GO TO 500
242  DRI=SQRT (MAX**2-(A*COS (PHI))/(2.0*3.1416))-RI
      DRC=SQRT (MAX**2-(A*COS (PHI))/(2.0*3.1416))-RO
      RT=RT+DRT/2.0
      RI=RI+DRI
      ROR=RO+DRO/2.0
      RO=RO+DRO
      DTI=(MAX-RI)/(COS (PHI))-YI
      DYO=(RO-RAX)/(COS (PHI))-YO
      YI=YI+DTI
      YO=YO+DYO

UU  COMPUTE ANGLE OF NOZZLE INNER WALL RELATIVE TO AXIS.
      MIN=ATAN (DYI/DXI)
      MIN=MIN*57.29578
246  DTHI=7.5*CFIM*((A*BN)/(1.0+R**2))*DX-THIM*
      ((1.131*DVBSQ)/VBSQ-DA/AN*DRT/PIH)
      THIM=(C*IM*HOL*VLSQ)/(6559.6*(1.0+R**2))
      THI=THI-DTHI
      DELI=1.286*THI
      DELSI=1.2857*THI
69  DTHO=7.5*CFIM*((A*BN)/(1.0+R**2))*DX-THOM*
      ((1.131*DVBSQ)/VBSQ-DA/AN*DRO/ROH)
      THOM=(C*OM*HOL*VLSQ)/(6559.6*(1.0+R**2))
      THO=THO-DTHO
      DELO=1.286*THO
      DELSO=1.2857*THO

UU  COMPUTE ANGLE OF NOZZLE OUTER WALL RELATIVE TO AXIS.
      WOF=ATAN (DYO/DX)
      WOF=57.29578*WOF

UU  COMPUTE MEAN VELOCITY (INCLUDING BOUNDARY LAYER).
251  VBD=VB*(1.0-((2.0*3.1416)/A)*
      ((C*THO+RI*THI))
      GO TO 61

UUUU  IF AREA LESS ZERO, NOZZLE IS CONVERGING AND THROAT HAS
      NOT JUST BEEN PASSED. IF NOT, NOZZLE THROAT MAY
      HAVE JUST BEEN PASSED.
61  IF (A-AT) 66,62,62

UU  ADVANCE STEP COUNT AND RETURN TO STEP 30 IF MAXIMUM
      STEP COUNT NOT REACHED.
62  NNS=NNS+1
      IF (NNS-NS) 30,260,260

UU  INITIALIZE TB AND COUNTERS AND GO TO STEP 140 TO
      COMPUTE BARRER QUANTITIES.
260  TB=TL
      NNS=0

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19450	19450
19460	19460
19470	19470
19480	19480
19490	19490
19500	19500
19510	19510
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19550	19550
19560	19560
19570	19570
19580	19580
19590	19590
19600	19600
19610	19610
19620	19620
19630	19630
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19670	19670
19680	19680
19690	19690
19700	19700
19710	19710
19720	19720
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19780	19780
19790	19790
19800	19800
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19890	19890
19900	19900
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19950	19950
19960	19960
19970	19970
19980	19980
19990	19990
20000	20000


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SZC=50
SM=SM
HOPR=RESC*RESZ
IF (HOPR) 3080,3085,3085
3080 CONTINUE
ASLI=RESZ
DSLI=JRN
SOJ=70
GO TO 1090
3085 CONTINUE
RESI=RESZ
SMJ=324
SOJ=520
3090 CONTINUE
SM=(SMJ+DSLI)/2.0
LJH=J
16000 CONTINUE
IF (ABS(SM)-SMIN) 1390,1390,1391
1390 CONTINUE
SM=-SIGN(SMIN,DP)
AM=1.0*(RESI)/(1.0+R)
BM=1.0*(SM)/(1.0+R)
VBSQ=1-VI/AM-1/2/BM
VBSQ=VBSQ+VBSQ/2.0
C
WRITE DIAGNOSTIC IF VBR NON-POSITIVE.
C
IF (VBSQ) 1250,1250,1251
1250 CONTINUE
WRITE (6,250) SM,AM,BM,V1,V2,VBSQ,VBSQ,R,VBSQ
LL=18
GO TO 1492
C
COMPUTE VBR
C
1251 VBR=1.0*(VBSQ)
RES=1.0*VI*ABS(SM)
IF (RES=-2000.0) 1259,1259,1259
1258 CD=2.0*RES
GO TO 71
1259 IF (RES=0.0) 1260,1260,1263
1263 IF (RES=-2000.0) 1261,1264,1264
1264 CD=0.45*RES
GO TO 71
1260 CD=24.0/RES
GO TO 71
1261 CD=EXP(1.271-0.8971*(ALOG(RES)+0.33417*
1*((ALOG(RES))**2)+.001041*((ALOG(RES))**3)))
71 CONTINUE
***** STATEMENT NO. 80 *****
C
OPTIMUM SM IS TO BE USED.
C
80 IF (RES=-1611) 1380,1380,1381
1380 CD=1.0
GO TO 1384
1381 IF (RES=-4202.11392,1382,1383
1382 CD=1.1174*LOG10((ALOG(RES))+.004129*((ALOG(RES))**2))
GO TO 1384
1383 CD=1.0
1384 SMLC=(CD*(34**21)/(VBSQ*CD)
SMLD=(SMLC*(34**21)/(VBSQ*(AM**2)))
SMLC=(SMLD*(34**21)/(SMLD*AM*(1.0+R)))
SMLC=(SMLC*(1.0+R))-1.0
SMLD=SMLC*((1.0+AM)/(1.0+R))+1.0+1.0
D=1.0
JHH 8/19/66
Z2Z=SMLD*SMLD*SM*SMLD*CD
SO=-SIGN(1.0,D**Z2Z)
C2=2.0*SMLD*SMLD
C2=SMLC*Z2Z/C2
IF (C2.LT.-J.C) C2=ABS(C2)

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      C2 = C2**0.33333
      SO = SO/C2
      SSM = SC/SM
      81 IF (ABS (SC-SM) - ESO) 1481, 1481, 1482
      1481 NIS=NMSO
      NMSO=0
      REMIND 16
      GO TO 42
C*****THIS IS TO BYPASS ITERATION COUNTER DURING PLOTTING IJH = 4
      1482 CONTINUE
      1482 NMSO = NMSO + 1
      16360 IF (NMSO-NSO) 1483, 1483, 1490
C
      STORE THIS VALUE OF DS1 AND WRITE DS1 ARRAY IF FULL.
      1483 CONTINUE
C*****DETERMINING AID FOR BINARY CUT CONVERGENCE
      DSPAT(KDS) = 22222222.
      DSPAT(KDS+1) = S*J1
      DSPAT(KDS+2) = SCJ1
      DSPAT(KDS+3) = SCJ2
      DSPAT(KDS+4) = SM
      DSPAT(KDS+5) = SO
      DSPAT(KDS+6) = SSM
      DSPAT(KDS+7) = DDLIN
      DSPAT(KDS+8) = SCJ2
      DSPAT(KDS+9) = RDLIN
      KDS = KDS + 10
      IF (KDS - 100) 16003, 16003, 1484
      1484 WRITE (16) DSPAT
      KDSCT = KDSCT + 1
      DO 1486 I=1, 100
      DSPAT(I) = 0.0
      1486 CONTINUE
      KDS=1
      16003 CONTINUE
      GO TO (3030, 1031, 3071) IJH
C*****END OF BINARY CUT CONVERGENCE ROUTINE
C
      NO CONVERGENCE ON DS - PRINT DIAGNOSTIC AND GET ALL
      PREVIOUS VALUES OF DS FROM TABE 15 AND TEMPORARY STORAGE IN
      DSRA1. PRINT THEM AND EXIT TO DIAGNOSTIC ROUTINE.
      1490 REMIND 16
      NST=72299
      WRITE (16, 3470)
      1491 IF (KDS-1) 1493, 1493, 1492
      1492 READ (16) DSRA1
      WRITE (6, 3491) (DSRA1(I), I=1, 100)
      KDSCT = KDSCT - 1
      GO TO 1491
      1493 WRITE (6, 3491) (DSRA1(I), I=1, KDS)
      KDS=1
      KDSCT=0
      DO 1495 I=1, 100
      DSPAT(I) = 0.0
      1495 CONTINUE
      LL=18
      REMIND 16
      GO TO 500
C
      OPTIMIZE DS AND DX.
      82 IF (NFO-1) 2382, 2381, 2382
      2381 DS=0.0
      S=SO
      NFO=0
      GO TO 93
      2382 DS=SO-SOF
      83 SOP=SO
      DX = (D/ (0.75*RHOG*ABS (SM) *SM*CD*V*NSQ))
      * (4633.04*DP* ((BH**2)*RHOL*DV*NSQ)/2.0)

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SEC5 1486
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SEC5 2600

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2      * (( (RN*BHOL*VBMSQ) / (1.0+R) ) * (( (2.0*SM*DT) / (1.0+R) ) - DS) )
100  DS1=75
      NST=9999
      VB=VBMSQ*DTBSSQ
      IF (VBMSQ) 100, 300, 301
300  WRITE (6,9100)
      LL=19
      GO TO 500
C
C      STORE **DPPOINT QUALITIES FOR NEXT END-POINT.
C
301  ENLN=ENL
      RHCLN=RHCL
      DELTN=TG-TL
      VLN=VM*VBS
      HLEN=HLA
      HLEN=HLCB
      RN=R
C
C      DETERMINE HEAT TRANSFER COEFFICIENT.  SEGMENTS ARE NOT
C      CONTINUOUS AT JUNCTIONS.
      IF (RN-1.0) 304, 304, 105
304  HN=(24.4*HKGM)/D
      GO TO 309
305  IF (RN-25.0) 306, 306, 107
306  HN=(300.0*CM*PHO:*(ABS(SN))**VBS*
      * ((2.2/RE**(.48/(3277(RN))))))
      GO TO 309
307  HN=(0.44*HKGM*(RN**-.6))/D
C
C      COMPUTE NEAR BOUNDARY-LAYER PARAMETERS FOR CIRCULAR NOZZLE
C      OR OUTER WALL OF ANNULAR NOZZLE.
308  DELN=180*DELN/2.0
      RDELN=10.246*DELN
      RDELN=(100.0*PHO*VL*DELN)/VILN
      IF (RDELN<1.0) 309, 309, 309
      CF=2.208/((ALOG10(RDELN)*.425)**2.594)
      IF (RDELN) 309, 309, 309
309  DELN=10.246*DELN
      RDELN=(100.0*PHO*VL*DELN)/VILN
      CF=2.208/((ALOG10(RDELN)*.425)**2.594)
30  NS1=10
      RETURN
2399  NS1=2199
      RETURN
500  NDIAG=1
      CALL OUTUT(NNN,LPRCT,NS50)
      NS1=110
      RETURN
9250  FORMAT(1H0.7X,15XNONPOSITIVE TBM/(1H ,3X,1P6R15.6))
9300  FORMAT(1H0.7X,14XNONPOSITIVE VB)
9490  FORMAT(1H0.7X,24XNO CONVERG ON DS,/)
9491  FORMAT(1H ,1P10E13.4)
3061  FORMAT(1H0.3XHLK* SYIN RESIDUES IN INTERVAL 3 TO ,1P12.4,
      14HFOR DP = ,E15.4)
      END
      SUBROUTINE SECT6
COMMON TZZZAY
DIMENSION TZZZAY(11200)
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTRF
IT MUST BE THE FIRST COMMON BLOCK
C
COMMON A , ABAR , ALAM , ALPHAB , ALPHA , ALPHAB ,
1  ALPHS , AM , AP , AS , AT , SA ,
2  BB , BETAB , BETAMB , BETA , BETAS , BM ,
3  CT , CAG1 , CAGMB , CAG , CAL13 ,
4  CBG1 , CAL , CASE , CBGMB , CBG , CBEND , COL
COMMON CDNS , CDP , CD , CPIN , CFIN , CFIN ,
5  CPM15 , CGBB , CGN , CGNS , CLMB , CLN ,
6  CLBS , CSAVE , D , D2 , DA , DATE ,
7

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318 IF (N) 318, 318, 320
319 LL=20
WRITE (6, 931H) HT (NN), TB, P
GO TO 500
C C C
COMPONENT A. PRINT DIAGNOSTIC IF SUFFICIENTLY CLOSE TO ZERO.
C C C
320 IF (ABS (1.0-H*PBB)-.0001) 321, 321, 323
321 WRITE (6, 932H)
LL=20
GO TO 500
C C C
COMPUTE PARTIAL PRESSURE OF COMPONENT A AND B AND
PRINT DIAGNOSTIC IF EITHER NONPOSITIVE.
C C C
323 PAB=(P-PEQB)/(1.-(H*PBB))
IF (PAB) 311, 322, 322
322 PBB=P-PAB
IF (PBB) 312, 9323, 9323
C C C
INTERPOLATE INTO MOLECULAR WEIGHT TABLES FOR A AND B
AT THIS TEMPERATURE AND PRESSURE.
C C C
9323 N=3
NN=3
CALL INTERP(WGB, N, TB, PAB)
IF (N) 325, 325, 327
325 LL=20
WRITE (6, 931H) HT (NN), TB, PAB
GO TO 500
327 N=4
NN=4
CALL INTERP(WGB, N, TB, PBB)
IF (N) 329, 329, 330
329 LL=20
WRITE (6, 931H) HT (NN), TB, PBB
GO TO 500
330 WRITE (6, 931H)
LL=20
GO TO 500
331 LL=20
WRITE (6, 931H)
LL=20
GO TO 500
332 LL=20
WRITE (6, 931H)
LL=20
GO TO 500
C C C
COMPUTE MEAN MOLECULAR WEIGHT.
C C C
330 WGB=((WAGB*PAB)+(WGB*PBB))/P
C C C
COMPUTE ALPHA AND BETA.
C C C
ALPHAB=((WAL/WBL)*H*PAB)/
(1.0+((WAL/WBL)-1.0)*H*PAB)
BETAB=((WGB*PBB)/(WGB*P))
C C C
CHECK DENOMINATOR OF EQUATION FOR R AND PRINT
DIAGNOSTIC IF SUFFICIENTLY CLOSE TO ZERO.
C C C
TR=(1.0-(1.0*RC)*ALPHAB)
INT=ABS (TR)
IF (INT-.001) 333, 335, 335
333 WRITE (6, 933H)
LL=20
GO TO 500
C C C
COMPUTE BARBED FLOW PARAMETERS.
C C C
335 ZP=(PC-(1.0*RC)*BETAB)/TR
DEMB=(P*BT/(1.0*PBB))-ENG
ZMB=RO*ENG
DEMBG=((ZMB*WAGB*PAB)/(WGB*P))-ZBAG
DEMBGB=((ZMB*WGB*PBB)/(WGB*P))-ZMBG

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      ENAGP=ENAG+DENAGB
      ENBGE=ENEG+DENBGB
      DTGB=(3-TG)/2.0
      TLNB=(TL+TN)/2.0
      TGNB=(TG+TN)/2.0
      PANG=(PA+PAB)/2.0
      PSNB=(PB+PBB)/2.0

      INTERPOLATE INTO SPECIFIC HEAT TABLES FOR
      CALNB - SPECIFIC HEAT - LIQUID A
      CBLNB - SPECIFIC HEAT - LIQUID B
      CAGNB - SPECIFIC HEAT - GAS A
      CBGNB - SPECIFIC HEAT - GAS B

      N=5
      NN=5
      CALL INTER(CALNB,N,TLNB,P)
      IF(N) 340,340,341
340  WRITE(6,9136) HT(NN),TLNB,P
      LL=25
      GO TO 500
341  N=6
      NN=6
      CALL INTER(CBLNB,N,TLNB,P)
      IF(N) 340,340,343
343  N=1
      NN=1
      CALL INTER(CAGNB,N,TGNB,PAB)
      IF(N) 344,344,345
344  LL=25
      WRITE(6,9136) HT(NN),TGNB,PAB
      GO TO 500
345  N=2
      NN=2
      CALL INTER(CBGNB,N,TGNB,PBB)
      IF(N) 346,346,347
346  LL=25
      WRITE(6,9136) HT(NN),TGNB,PBB
      GO TO 500
347  ALPHA=(ALTA+ALTB)/2.0
      BETAB=(BETA+PBTA)/2.0
      CENB=(ALBNB+CALNB)*(1.0-ALPHAB)+CBLNB
      CGNB=(1.0-BETAB)*CAGNB+CBGNB
      DTLE=(-1.0/(CENB+CGNB))*((CENB+CGNB)*DTGB+
      1  ((DENAGB/ENGB)*(HLA+CGA*(TG-TL)))+
      2  ((DENAGB/ENGB)*(HLB+CBG*(TG-TL)))+
      3  ((DENGB/ENGB)*(HLB+CBG*(TG-TL)))+
      4  ((DENGB/ENGB)*(((VG**2)-(VL**2))/50072.9)))

      TEST FOR CONVERGENCE ON TB.
      IF(ABS((TB-TL)-DTLE))-ES) 141,141,350
350  NNE=NNB+1
      IF(NNE-NB) 351,351,354
      SAVE THIS VALUE OF TB FOR POSSIBLE NON-CONVERGENCE.
351  TB=TL+DTLE
      TBAI(KTB)=TB
      KTB=KTB+1
      IF(KTB-100) 140,140,352
352  WRITE(14) TBAI
      DO 353 I=1,100
      TBAI(I)=0.0
353  CONTINUE
      KTB=1
      KTBCT=KTBCT+1
      GO TO 140

      NO CONVERGENCE ON TB AND ITERATION MAXIMUM REACHED.
      PRINT DIAGNOSTIC AND ALL VALUES OF TB FOUND.

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SEC6 1600
SEC6 1610
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SEC6 2290
SEC6 2300

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354 WRITE (6,9452)
359 IF (KTBC) 403,403,400
400 READ (14) TBAY1
WRITE (6,9491) (TBAY1(I), I=1,100)
KTBC=KTBC-1
GO TO 359
403 WRITE (6,9491) (TBAY1(I), I=1,KTBC)
C
C CLEAR TB SAVE ARRAY.
C
KTBC=1
KTBC=0
DO 405 I=1,100
TBAY1(I) = 0.0
405 CONTINUE
LL=25
GO TO 500
C STORE NUMBER OF ITERATIONS REQUIRED TO CONVERGE ON TB.
C
C ***** STATEMENT NO. 140 *****
C
141 NIP=400
NND=1
C
C COMPUTE DENSITY OF GAS MIXTURE.
C
RHOGB= (WGB*P)/(10.732*TB)
C
C INTERPOLATE FOR DENSITIES OF LIQUIDS A AND B.
C
N=10
NN=10
CALL INTFP (ROALB,N,TB,P)
LL=27
IF (N) 319,319,360
360 N=11
NN=11
CALL INTRP (ROBLB,N,TB,P)
LL=27
IF (N) 319,319,363
C
C COMPUTE DENSITY OF LIQUID MIXTURE.
C
363 RHOLB=1.0/((ALPHA*ROALB)+(1.0-ALPHA)/(ROBLB))
ABAR=((144.0*ENGL)*((1.0/RHOLB+3B/RHOLB))/VR
RVE=1.0/((RHOGB+RHOGB)/VR
Q=(RHOLB*ABAR+RVE)/(144.0*ENLB)
C
C INTERPOLATE INTO VISCOSITY TABLES FOR LIQUID A AND B.
C
N=14
NN=14
CALL INTBP (VIALB,N,TB,P)
LL=27
IF (N) 319,319,366
366 N=15
NN=15
CALL INTRP (VIBLB,N,TB,P)
LL=27
IF (N) 363,363,370
369 GO TO 319
370 VILB=ALPHA*VIALB+(1.0-ALPHA)*VIBLB
REF=(8617.2*ENLB)/(VILB*(307+1847))
C ENTER CURRENT VALUES OF A AND P INTO A VS P TABLE
C
529 NSTMT=529
RETURN
C
C *****
C
C FIND GOOD APPROXIMATION FOR DS1 TO USE IN SUCCESSIVE

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      SGN=SGN/10.0
      ISL=ISL+1
      IF (ISL-5) 2403,2403,5060
12017 CONTINUE
      DSAVE=DSRES
      DSX2=DS1
      GO TO 2409

      EVALUATE DS FOR GIVEN DS1.  TEST FOR CONVERGENCE
      #####

5000 CONTINUE
      SM=S*DS1/2.0
      AM=1.0+(R*SM)/(1.0+R)
      BM=1.0-SM/(1.0+R)
      DVBSQ=(-V1/AM)-(V2/BM)
      VBNSQ=VBSSQ+DVBSQ/2.0
      IF (V1=0) GO TO 5001,5001,5001

6000 CONTINUE
      WRITE (6,7001) SM,S,DS1,AM,B,DM,DVBSQ,V1,V2,VBNSQ,VBN
5001 CONTINUE
      VBNSQ=VBNSQ/(VUNSQ)
      REM=1+VBNSQ*(S-1)
      IF (REM=-.000001) 5015,5015,5005
5005 IF (REM=-.01) 5020,5020,5010
5010 IF (REM=-.000001) 5030,5025,5025
5015 CD=2.4E-7
      GO TO 5015

5020 CD=24.3E-7
      GO TO 5015

5025 CD=0.4E-7
      GO TO 5015

5030 CD=EXP(3.271-0.8843*(ALOG(REM)+40.03417*
111(ALOG(REM))**2)+.001344*(ALOG(REM))**3))
5035 CONTINUE
      DSXP=((51*ADS(SM)*SM*CD)/DM)+(S2/(DM*VUNSQ))
      I=((S1-1)*DVBSQ)/(VUNSQ)+S4*SM
      DSRP=57E-27
      IF (ABS(DSRP)-DS) 91,91,5040
5040 DSR1=(DS)-DS1
      DSR4V(KDS+1)=DSRES
      KDS=KDS+1
      IF (KDS-100) 5055,5055,5045
5045 WRITE (16) DSR4V
      KDSCT=KDSCT+1
      GO 5060 L=1,100
5050 DSR4V(L)=0.0
      KDS=1
5055 VBNSQ=VBNSQ+1
      IF (VBNSQ-NDS) 5070,5070,5060
5060 WRITE (6,5060) DSX1,DSX2,DS1,DSRES1,DSRES2,DSRES
      LL=18
      GO TO 1490
5070 CONTINUE
      GO TO (2401,2402,2404,12411,2405,2407,2409),ISL
#####
      COMPUTE NEXT APPROXIMATION TO DS.

91 NIS=KNNS
      IRLX=1
      DS=DS1
      DSAVE=DS1
      KNS=0
      GO TO 100
1490 NSTXT=1490
      RETURN
100 NSTXT=100
      RETURN
500 NDIAG=1
      CALL OUTPUT(NNN,LPGCT,NGRO)

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NSTMT=110
RETURN
5065 FORMAT(1H0.7X,51HNO DS CONVERGENCE.  COULD DETECT NO SIGN CHANGE
1IN, 3HRESIDUALS OF F(DS).  VALUES FOLLOW / (10X,1P6E15.6))
9001 FORMAT(1H0.10X,50HNEGATIVE VERNOS,32CT6 / (1X,1P6E15.6))
9136 FORMAT(1H0.7X,17HOUTSIDE RANGE OF , A6.7H TABLE,.5X,
14HT = , 1P212.4, 5X,4H2 = F12.4)
9310 FORMAT(1H0.7X,12HNEGATIVE PAB)
9312 FORMAT(1H0.7X,12HNEGATIVE PAB)
9318 FORMAT(1H0.7X,17HOUTSIDE RANGE OF , A6.7H TABLE,.5X,
14HT = , 1P212.4, 5X,4H2 = F12.4)
9320 FORMAT(1H0.7X,12HNEGATIVE PAB)
9333 FORMAT(1H0.7X,14HNO POSITIVE EN)
9352 FORMAT(1H0.7X,24HCOULD NOT CONVERGE ON P3)
9491 FORMAT(1H,1P10E11.4)
END

SUBROUTINE THROAT
COMMON TZZZAY(11230)
DIMENSION TZZZAY(11230)
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTRD
IT MUST BE THE FIRST COMMON BLOCK
C
COMMON A, ABAR, ALAN, ALPHAB, ALPHA, ALPHNB,
1 ALPHS, AB, ABAR, ABAR, ABAR, ABAR, ABAR,
2 C, C1, C2, C3, C4, C5, C6, C7, C8, C9, C10,
3 CSB1, CAL, CAPP, CAPP, CAPP, CAPP, CAPP,
COMMON CCM1, CCM2, CCM3, CCM4, CCM5, CCM6, CCM7, CCM8, CCM9, CCM10,
6 CCM11, CCM12, CCM13, CCM14, CCM15, CCM16, CCM17, CCM18, CCM19, CCM20,
7 CCM21, CCM22, CCM23, CCM24, CCM25, CCM26, CCM27, CCM28, CCM29, CCM30,
8 CCM31, CCM32, CCM33, CCM34, CCM35, CCM36, CCM37, CCM38, CCM39, CCM40,
9 CCM41, CCM42, CCM43, CCM44, CCM45, CCM46, CCM47, CCM48, CCM49, CCM50,
COMMON DLOS, DELTA, DELTA, DELTA, DELTA, DELTA,
1 DLSIS, DLSIS, DLSIS, DLSIS, DLSIS, DLSIS,
2 DSI, DSI, DSI, DSI, DSI, DSI,
3 DSXAX, DSXAX, DSXAX, DSXAX, DSXAX, DSXAX,
COMMON DSSES, DSSES, DSSES, DSSES, DSSES, DSSES,
6 DSSES, DSSES, DSSES, DSSES, DSSES, DSSES,
7 DSSES, DSSES, DSSES, DSSES, DSSES, DSSES,
8 DSSES, DSSES, DSSES, DSSES, DSSES, DSSES,
9 DSSES, DSSES, DSSES, DSSES, DSSES, DSSES,
COMMON HGT1, HGT1, HGT1, HGT1, HGT1, HGT1,
1 HGT1, HGT1, HGT1, HGT1, HGT1, HGT1,
2 HGT1, HGT1, HGT1, HGT1, HGT1, HGT1,
3 HGT1, HGT1, HGT1, HGT1, HGT1, HGT1,
4 HGT1, HGT1, HGT1, HGT1, HGT1, HGT1,
COMMON K2, K2, K2, K2, K2, K2,
6 K2, K2, K2, K2, K2, K2,
7 K2, K2, K2, K2, K2, K2,
8 K2, K2, K2, K2, K2, K2,
9 K2, K2, K2, K2, K2, K2,
COMMON LSW, LSW, LSW, LSW, LSW, LSW,
1 LSW, LSW, LSW, LSW, LSW, LSW,
2 LSW, LSW, LSW, LSW, LSW, LSW,
3 LSW, LSW, LSW, LSW, LSW, LSW,
4 LSW, LSW, LSW, LSW, LSW, LSW,
COMMON PAS, PAS, PAS, PAS, PAS, PAS,
6 PAS, PAS, PAS, PAS, PAS, PAS,
7 PAS, PAS, PAS, PAS, PAS, PAS,
8 PAS, PAS, PAS, PAS, PAS, PAS,
9 PAS, PAS, PAS, PAS, PAS, PAS,
COMMON PLS, PLS, PLS, PLS, PLS, PLS,
6 PLS, PLS, PLS, PLS, PLS, PLS,
7 PLS, PLS, PLS, PLS, PLS, PLS,
8 PLS, PLS, PLS, PLS, PLS, PLS,
9 PLS, PLS, PLS, PLS, PLS, PLS,
COMMON R1, R1, R1, R1, R1, R1,
6 R1, R1, R1, R1, R1, R1,
7 R1, R1, R1, R1, R1, R1,
8 R1, R1, R1, R1, R1, R1,
9 R1, R1, R1, R1, R1, R1,
COMMON S1, S1, S1, S1, S1, S1,
6 S1, S1, S1, S1, S1, S1,
7 S1, S1, S1, S1, S1, S1,
8 S1, S1, S1, S1, S1, S1,
9 S1, S1, S1, S1, S1, S1,
COMMON S1, S1, S1, S1, S1, S1,
6 S1, S1, S1, S1, S1, S1,
7 S1, S1, S1, S1, S1, S1,
8 S1, S1, S1, S1, S1, S1,
9 S1, S1, S1, S1, S1, S1,
COMMON T, T, T, T, T, T,
6 T, T, T, T, T, T,
7 T, T, T, T, T, T,
8 T, T, T, T, T, T,
9 T, T, T, T, T, T,
COMMON THOB, THOB, THOB, THOB, THOB, THOB,
6 THOB, THOB, THOB, THOB, THOB, THOB,
7 THOB, THOB, THOB, THOB, THOB, THOB,
8 THOB, THOB, THOB, THOB, THOB, THOB,
9 THOB, THOB, THOB, THOB, THOB, THOB

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6		CFORS	CGMB	CGM	CGMS	CL4B	CL4	DIAG0110
7		CLMS	CSAVE	D	D2	DA	DATC	DIAG0110
8		DD	DELSI	DELI	DELIS	DELOM	DELO	DIAG0110
9		DELOS	DELSQ	DELFM	DELFM	DELT	DELV2	DIAG0110
1	COMMON	DELAGB	DEMBG	DEMBG	DEMBG	DEMBG	DEMBG	DIAG0110
2		DISTS	DLSOS	D	D1	D	D	DIAG0110
3		DPI	DRO	D	D1	D	D	DIAG0110
4		DSMAX	DSMIN	DSRAY1	DSRAY	DSRAY1	DSRAY2	DIAG0110
5	COMMON	DSRPS	DS	DSX1	DSX2	DTM1	DTM2	DIAG0110
6		DTG	DTHT	DTNO	DTL1	DTL8	DTL	DIAG0110
7		DYO	DVBSQ	DVBSQ	DVLSQ	D	D	DIAG0110
8		ENBG	ENGB	ENB	ENAG8	ENAG	ENAG8	DIAG0110
9		FML	FMLS	ENT	ESQ	FLERP	FLERP	DIAG0110
1	COMMON	H	HJT	HVAG	HVAG	HKGM1	HKGM2	DIAG0110
2		HKGM	HKGNS	HLA1	HLAN	HLA	HLAS	DIAG0110
3		HLB1	HLBN	HLR	HLR	HL	HL	DIAG0110
4		HLP2	HT	IACV	ISL4	ISLX	ISL	DIAG0110
5		IJH	IJT	IJCT	ISL4	ISLX	ISL	DIAG0110
6	COMMON	K2	KAP	KDSCT	KDS	KL	KN1	DIAG0110
7		KR	KTBC	KTB	KXP1	L	L	DIAG0110
8		L1	L2	LCT1	LCT2	LL	LCTCT	DIAG0110
9		LSW	LSW	MPDO	MP	MGFO	MG	DIAG0110
1	COMMON	MCP	MFT	NACV	NH	NCLK	NCLT	DIAG0110
2		NDIAG	NDS	NH	NH	NH	NH	DIAG0110
3		NH300	NH300	NH300	NH300	NH300	NH300	DIAG0110
4		PAG	PAG	PAG	PAG	PAG	PAG	DIAG0110
5	COMMON	PL1	PL1	PL1	PL1	PL1	PL1	DIAG0110
6		RC	RC	RC	RC	RC	RC	DIAG0110
7		REHOG	REHOG	REHOG	REHOG	REHOG	REHOG	DIAG0110
8	COMMON	RIN	RIN	RIN	RIN	RIN	RIN	DIAG0110
9		ROBL	ROBL	ROBL	ROBL	ROBL	ROBL	DIAG0110
1		RVB	RVB	RVB	RVB	RVB	RVB	DIAG0110
2		GJ	GJ	GJ	GJ	GJ	GJ	DIAG0110
3	COMMON	SIGS	SIGS	SIGS	SIGS	SIGS	SIGS	DIAG0110
4		SMLE	SMLE	SMLE	SMLE	SMLE	SMLE	DIAG0110
5		SOP	SOP	SOP	SOP	SOP	SOP	DIAG0110
6		T	T	T	T	T	T	DIAG0110
7	COMMON	TG	TG	TG	TG	TG	TG	DIAG0110
8		THOH	THOH	THOH	THOH	THOH	THOH	DIAG0110
9		TL4B	TL4B	TL4B	TL4B	TL4B	TL4B	DIAG0110
1		TATMS	TATMS	TATMS	TATMS	TATMS	TATMS	DIAG0110
2		VBO	VBO	VBO	VBO	VBO	VBO	DIAG0110
3	COMMON	VIALB	VIALB	VIALB	VIALB	VIALB	VIALB	DIAG0110
4		VIGMS	VIGMS	VIGMS	VIGMS	VIGMS	VIGMS	DIAG0110
5		VL	VL	VL	VL	VL	VL	DIAG0110
6		VAG	VAG	VAG	VAG	VAG	VAG	DIAG0110
7		WHL	WHL	WHL	WHL	WHL	WHL	DIAG0110
8	COMMON	XUS	XUS	XUS	XUS	XUS	XUS	DIAG0110
9		XINT	XINT	XINT	XINT	XINT	XINT	DIAG0110
1		YIS	YIS	YIS	YIS	YIS	YIS	DIAG0110
2		YIS	YIS	YIS	YIS	YIS	YIS	DIAG0110

DIMENSION PL(10)
 DIMENSION SGRAY(75)
 DIMENSION CASE(3), DID(8), X2(2,75), PL1(10)
 1 C(10), R(2,75), XXA(8), DSRAY(100), TBRAY(100), TBRAY(100)
 2 (100), SGRAY(75), L(10), DENT(8)
 EQUIVALENCE (NDRAY(1), SGRAY(1))
 IS (NSTRT-497) 1,497,2
 1 CALL DIME
 2 IF (NSTRT-529) 1,529,1
 497 NSTRT=999
 1 I=(NDIAG-1) 10,498,10
 498 DO 499 I=1,75
 DRAY(I)=99999.
 499 CONTINUE
 LSW=0

```

GO TO(501,502,503,504,505,506,507,508,509,510,511,512,513,514,
1 515,516,517,518,519,520,521,522,523,524,525,526,527,528,
2 529),LL
DIAG 0850
DIAG 0860
DIAG 0870
DIAG 0880
DIAG 0890
DIAG 0900
DIAG 0910
DIAG 0920
DIAG 0930
DIAG 0940
DIAG 0950
DIAG 0960
DIAG 0970
DIAG 0980
DIAG 0990
DIAG 1000
DIAG 1010
DIAG 1020
DIAG 1030
DIAG 1040
DIAG 1050
DIAG 1060
DIAG 1070
DIAG 1080
DIAG 1090
DIAG 1100
DIAG 1110
DIAG 1120
DIAG 1130
DIAG 1140
DIAG 1150
DIAG 1160
DIAG 1170
DIAG 1180
DIAG 1190
DIAG 1200
DIAG 1210
DIAG 1220
DIAG 1230
DIAG 1240
DIAG 1250
DIAG 1260
DIAG 1270
DIAG 1280
DIAG 1290
DIAG 1300
DIAG 1310
DIAG 1320
DIAG 1330
DIAG 1340
DIAG 1350
DIAG 1360
DIAG 1370
DIAG 1380
DIAG 1390
DIAG 1400
DIAG 1410
DIAG 1420
DIAG 1430
DIAG 1440
DIAG 1450
DIAG 1460
DIAG 1470
DIAG 1480
DIAG 1490
DIAG 1500
DIAG 1510
DIAG 1520
DIAG 1530
DIAG 1540
DIAG 1550
DIAG 1560

CCCCC
DIAGNOSTIC OUTPUT. ENTRY IS MADE TO SET CERTAIN
SET OF OUTPUT VALUES INTO THE ARRAY DRAY DEPENDNG
ON THE VALUE OF LL, WHICH IS SET IN VARIOUS POINTS THROUGHOUT
THE MAIN PROGRAM.
529 DRAY(54)=RPF
NSTA=0000
DRAY(45)=VILB
528 DRAY(53)=Q
DRAY(34)=AAR
DRAY(47)=HHOLB
527 DRAY(46)=THOGB
526 DRAY(39)=TU
525 DRAY(44)=CNLB
DRAY(43)=ENGU
DRAY(37)=RU
524 DRAY(42)=BETAB
DRAY(41)=ALPHAB
DRAY(50)=WGB
DRAY(49)=WUTB
523 DRAY(48)=AAB
522 DRAY(52)=TAB
521 DRAY(51)=PAB
520 DRAY(73)=CEIN
DRAY(69)=RDELIN
DRAY(71)=TILN
DRAY(61)=CJN
DRAY(53)=DELON
DRAY(36)=NY
DRAY(35)=CU
519 DRAY(34)=RBN
518 DRAY(63)=VBJ
DRAY(64)=JON
DRAY(59)=DELLO
DRAY(58)=DELLO
DRAY(57)=THO
DRAY(62)=TJON
DRAY(65)=DELLO
DRAY(66)=THI
DRAY(56)=WIT
517 DRAY(51)=A
DRAY(12)=D
DRAY(24)=SIT
516 DRAY(34)=RA
DRAY(9)=VL
DRAY(4)=VG
515 DRAY(1)=X
DRAY(10)=CLN
DRAY(21)=CJN
DRAY(31)=VILN
DRAY(32)=VILN
DRAY(33)=VILN
DRAY(34)=VILN
DRAY(35)=VILN
GO TO 1511
514 DRAY(13)=HKTN
DRAY(32)=VILN
513 DRAY(31)=VILN
DRAY(29)=CJN
DRAY(30)=CLN
512 DRAY(64)=VBO
DRAY(54)=RBN
DRAY(54)=DELLO
DRAY(54)=DELLO
DRAY(57)=THO
DRAY(62)=TJON
DRAY(68)=DELLO
DRAY(67)=THI
DRAY(66)=THI

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DPAT(65)=WIM
DPAT(6)=A
DPAT(12)=D
DPAT(24)=SIG
DPAT(14)=PA
DPAT(9)=VL
DPAT(1)=VG
DPAT(1)=X
1511 DPAT(55)=Y0
DPAT(54)=YI
511 DPAT(11)=S
DPAT(4)=VB
DPAT(10)=VS
DPAT(5)=A
510 DPAT(27)=HL0
DPAT(25)=HLA
509 DPAT(11)=RV
DPAT(14)=ENL
DPAT(17)=FMS
DPAT(23)=RMDL
508 DPAT(19)=RHOG
DPAT(3)=R
507 DPAT(16)=DETA
DPAT(15)=ALPHA
DPAT(23)=WG
DPAT(22)=WEG
506 DPAT(21)=WAG
505 DPAT(25)=PU
504 DPAT(34)=PA
503 DPAT(7)=TL
502 DPAT(6)=TG
501 DPAT(2)=F
NDPAT(72)=WNA
NDPAT(73)=NIS
NDPAT(74)=NIB
IF (NDPAT-1) 540,550,540
540 WRITE (11) DPAT
NNN=NN+1
DPAT(12)=7.0
WRITE (11) (DPAT(J),J=1,10)
C
GO TO 63
C
DIAGNOSTIC OUTPUT - VARIABLES NOT YET COMPUTED THIS
(ITERATION ARE SET TO 1055. PRINT OUT NINE LINES
OF OUTPUT.
C
550 WRITE (6,1001) (DPAT(I),I=1,9)
551 WRITE (6,1002) (DPAT(I),I=10,18)
552 WRITE (6,1003) (DPAT(I),I=19,27)
553 WRITE (6,1004) (DPAT(I),I=28,36)
554 WRITE (6,1005) (DPAT(I),I=37,45)
555 WRITE (6,1006) (DPAT(I),I=46,54)
556 WRITE (6,1007) (DPAT(I),I=55,63)
557 WRITE (6,1008) (DPAT(I),I=64,71)
558 CONTINUE
WRITE (6,1009) NNA,NTS,NIB
10 NSTAT=0
RETURN
63 NSTAT=63
RETURN
C
FORMAT STATEMENTS FOR 9-LINE PERMANENT OUTPUT.
1001 FORMAT (1H,11X,1PX,1PY,1PZ,1PH,11X,1HR,12X,2HVN,11X,1HA,12X,2HTG,12X,
12HIL,12X,2HVG,12X,2HVL,/,5X,2FO6.4)
1002 FORMAT (15,12X,2HVS,11X,1HS,11X,1HS,12X,2HPV,12X,2HFA,9X,
15HALL,11X,1HRR,11X,2HRS,12X,2HRL,/,5X,2E14.4)
1003 FORMAT (1H,11X,3HPO,12X,2HGL,11X,1HWAG,11X,1HWSG,12X,2HWG,
12X,2HFA,12X,2HVS,12X,2HIL,12X,2HGL,/,5X,2E14.4)
1004 FORMAT (1H,9X,5H547HA,11X,1HCGH,11X,3HCL,10X,4HVIGN,10X,
14HVLH,11X,1HJGH,11X,1HJER,11X,1HJCH,12X,2HHR,/,5X,1E14.4)

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0005 FORNAT (1H0,12X,2HWP,12X,2HAB,12X,2HTR,11X,3HVP,9X,5HALPHB,
19X,5HDS,11X,3HNS,11X,3HLS,12X,4HVL,6X,5,9F14.4)
0006 FORNAT (1H0,10X,4HACG,10X,4HBL,10X,4HAG,10X,4HGB,
111X,3HAC,11X,3HAB,11X,3HBP,11X,3HBN,10X,3HAP,10X,3F14.4)
0007 FORNAT (1H0,12X,2HWP,11X,3HAC,11X,3HBN,10X,4HDEL,6X,5HDEL=0,
19X,5HRCO,12X,4HCPG,10X,4HCT,10X,3HVP,10X,3F14.4)
0008 FORNAT (1H0,12X,2HWP,11X,3HAC,11X,3HBN,10X,4HDEL,9X,5HDEL=1,9X,
15HRCO,10X,4HCPG,10X,4HCT,10X,3F14.4)
0009 FORNAT (1H1,10X,3HNS,9X,3HNL,9X,3HNL,9X,3HNL,12X,
C1032 FORNAT (1H1,120X,5HAPG,101X,15HDEL TWO-PIASE, 3(6X,16))
C 1 26H TWO-COMPONENT NOZZLE FLOW.//)
END
SUBROUTINE WRITE (DVGSG,DPL,RIOLN,JVLGJ,DETAG,DEMBG,
1 DELV2,DG,HAI,CAGI,DELT,HBL,CBGI,SM,AN,BM,DX,DS)
WRITE (5,111) DVGSG,DPL,RIOLN,DVLGJ,DETAG,DEMBG,
1 DELV2,DG,HAI,CAGI,DELT,HBL,CBGI,SM,AN,BM,DX,DS
RETURN
9231 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
15HDETAG,12X,5HDEMBG,12X,5HDELV2,11X,12F17.6,11X,3HAC,14X,
1 3HAI,14X,4HACGI,11X,4HBL,14X,3HBL,14X,4HCGI,11X,
1 15X,2HNS,11X,3HNL,6X,3F14.4)
2 14X,2HAB,11X,3HBP,15X,2HDS,11X,3F17.6)
END
SUBROUTINE OUTPUT (N,LPG,MCGO)
C SUBROUTINE TO READ SAVED PRINT LINES FROM LOGICAL
C TAPE 27 AND PRINT THEM ON FORTRAN OUTPUT TAPE.
DIMENSION DRAY (75),NDRAY (75)
EQUIVALENCE (DRAY (1),NDRAY (1))
REWIND 11
1
1 IF (N) 99,39,2
2 READ (11)DRAY
IF (L) 5,1
4 WRITE (6,101) LG
LPG=LPG+1
L=L+1
GO TO 6
5 WRITE (6,799)
L=L+1
6 WRITE (6,1001) (DRAY (I),I=1,75)
WRITE (6,1012) (DRAY (I),I=25,100)
WRITE (6,1003) (DRAY (I),I=25,100)
WRITE (6,1004) (DRAY (I),I=25,100)
WRITE (6,1005) (DRAY (I),I=25,100)
WRITE (6,1006) (DRAY (I),I=25,100)
WRITE (6,1007) (DRAY (I),I=25,100)
IF (MCGO) 7,8
7 WRITE (6,1008) (DRAY (I),I=64,71)
8 CONTINUE
9 WRITE (6,1009) (NDRAY (I),I=72,74)
N=N-1
GO TO 1
39 REWIND 11
3999 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
C1032 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
15HDETAG,12X,5HDEMBG,12X,5HDELV2,11X,12F17.6,11X,3HAC,14X,
1 3HAI,14X,4HACGI,11X,4HBL,14X,3HBL,14X,4HCGI,11X,
1 15X,2HNS,11X,3HNL,6X,3F14.4)
2 14X,2HAB,11X,3HBP,15X,2HDS,11X,3F17.6)
END
SUBROUTINE OUTPUT (N,LPG,MCGO)
C SUBROUTINE TO READ SAVED PRINT LINES FROM LOGICAL
C TAPE 27 AND PRINT THEM ON FORTRAN OUTPUT TAPE.
DIMENSION DRAY (75),NDRAY (75)
EQUIVALENCE (DRAY (1),NDRAY (1))
REWIND 11
1
1 IF (N) 99,39,2
2 READ (11)DRAY
IF (L) 5,1
4 WRITE (6,101) LG
LPG=LPG+1
L=L+1
GO TO 6
5 WRITE (6,799)
L=L+1
6 WRITE (6,1001) (DRAY (I),I=1,75)
WRITE (6,1012) (DRAY (I),I=25,100)
WRITE (6,1003) (DRAY (I),I=25,100)
WRITE (6,1004) (DRAY (I),I=25,100)
WRITE (6,1005) (DRAY (I),I=25,100)
WRITE (6,1006) (DRAY (I),I=25,100)
WRITE (6,1007) (DRAY (I),I=25,100)
IF (MCGO) 7,8
7 WRITE (6,1008) (DRAY (I),I=64,71)
8 CONTINUE
9 WRITE (6,1009) (NDRAY (I),I=72,74)
N=N-1
GO TO 1
39 REWIND 11
3999 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
C1032 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
15HDETAG,12X,5HDEMBG,12X,5HDELV2,11X,12F17.6,11X,3HAC,14X,
1 3HAI,14X,4HACGI,11X,4HBL,14X,3HBL,14X,4HCGI,11X,
1 15X,2HNS,11X,3HNL,6X,3F14.4)
2 14X,2HAB,11X,3HBP,15X,2HDS,11X,3F17.6)
END
SUBROUTINE OUTPUT (N,LPG,MCGO)
C SUBROUTINE TO READ SAVED PRINT LINES FROM LOGICAL
C TAPE 27 AND PRINT THEM ON FORTRAN OUTPUT TAPE.
DIMENSION DRAY (75),NDRAY (75)
EQUIVALENCE (DRAY (1),NDRAY (1))
REWIND 11
1
1 IF (N) 99,39,2
2 READ (11)DRAY
IF (L) 5,1
4 WRITE (6,101) LG
LPG=LPG+1
L=L+1
GO TO 6
5 WRITE (6,799)
L=L+1
6 WRITE (6,1001) (DRAY (I),I=1,75)
WRITE (6,1012) (DRAY (I),I=25,100)
WRITE (6,1003) (DRAY (I),I=25,100)
WRITE (6,1004) (DRAY (I),I=25,100)
WRITE (6,1005) (DRAY (I),I=25,100)
WRITE (6,1006) (DRAY (I),I=25,100)
WRITE (6,1007) (DRAY (I),I=25,100)
IF (MCGO) 7,8
7 WRITE (6,1008) (DRAY (I),I=64,71)
8 CONTINUE
9 WRITE (6,1009) (NDRAY (I),I=72,74)
N=N-1
GO TO 1
39 REWIND 11
3999 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
C1032 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
15HDETAG,12X,5HDEMBG,12X,5HDELV2,11X,12F17.6,11X,3HAC,14X,
1 3HAI,14X,4HACGI,11X,4HBL,14X,3HBL,14X,4HCGI,11X,
1 15X,2HNS,11X,3HNL,6X,3F14.4)
2 14X,2HAB,11X,3HBP,15X,2HDS,11X,3F17.6)
END
SUBROUTINE OUTPUT (N,LPG,MCGO)
C SUBROUTINE TO READ SAVED PRINT LINES FROM LOGICAL
C TAPE 27 AND PRINT THEM ON FORTRAN OUTPUT TAPE.
DIMENSION DRAY (75),NDRAY (75)
EQUIVALENCE (DRAY (1),NDRAY (1))
REWIND 11
1
1 IF (N) 99,39,2
2 READ (11)DRAY
IF (L) 5,1
4 WRITE (6,101) LG
LPG=LPG+1
L=L+1
GO TO 6
5 WRITE (6,799)
L=L+1
6 WRITE (6,1001) (DRAY (I),I=1,75)
WRITE (6,1012) (DRAY (I),I=25,100)
WRITE (6,1003) (DRAY (I),I=25,100)
WRITE (6,1004) (DRAY (I),I=25,100)
WRITE (6,1005) (DRAY (I),I=25,100)
WRITE (6,1006) (DRAY (I),I=25,100)
WRITE (6,1007) (DRAY (I),I=25,100)
IF (MCGO) 7,8
7 WRITE (6,1008) (DRAY (I),I=64,71)
8 CONTINUE
9 WRITE (6,1009) (NDRAY (I),I=72,74)
N=N-1
GO TO 1
39 REWIND 11
3999 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
C1032 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,
15HDETAG,12X,5HDEMBG,12X,5HDELV2,11X,12F17.6,11X,3HAC,14X,
1 3HAI,14X,4HACGI,11X,4HBL,14X,3HBL,14X,4HCGI,11X,
1 15X,2HNS,11X,3HNL,6X,3F14.4)
2 14X,2HAB,11X,3HBP,15X,2HDS,11X,3F17.6)
END
SUBROUTINE OUTPUT (N,LPG,MCGO)
C SUBROUTINE TO READ SAVED PRINT LINES FROM LOGICAL
C TAPE 27 AND PRINT THEM ON FORTRAN OUTPUT TAPE.
DIMENSION DRAY (75),NDRAY (75)
EQUIVALENCE (DRAY (1),NDRAY (1))
REWIND 11
1
1 IF (N) 99,39,2
2 READ (11)DRAY
IF (L) 5,1
4 WRITE (6,101) LG
LPG=LPG+1
L=L+1
GO TO 6
5 WRITE (6,799)
L=L+1
6 WRITE (6,1001) (DRAY (I),I=1,75)
WRITE (6,1012) (DRAY (I),I=25,100)
WRITE (6,1003) (DRAY (I),I=25,100)
WRITE (6,1004) (DRAY (I),I=25,100)
WRITE (6,1005) (DRAY (I),I=25,100)
WRITE (6,1006) (DRAY (I),I=25,100)
WRITE (6,1007) (DRAY (I),I=25,100)
IF (MCGO) 7,8
7 WRITE (6,1008) (DRAY (I),I=64,71)
8 CONTINUE
9 WRITE (6,1009) (NDRAY (I),I=72,74)
N=N-1
GO TO 1
39 REWIND 11
3999 FORNAT (1H0,12X,5HVGSG,14X,3HDTL,12X,5HRHOLN,12X,5HVLGJ,12X,

```


	R=1	TABLE 0470
	IF (L-15) 4, 4, 27	TABLE 0440
C	THIS CHANGE TO MAKE THE MAX DIMENSION CONSISTENT WITH THE DIMENSION	TABLE 0400
15	IF (I-4) 16, 17, 17	TABLE 0500
16	L=1	TABLE 0510
	I = I + 1	TABLE 0520
	K=1	TABLE 0530
	GO TO 4	TABLE 0540
27	WRITE (6, 927) I	TABLE 0550
	GO TO 17	TABLE 0560
17	CONTINUE	TABLE 0570
	REWIND 12	TABLE 0540
	WRITE (12) XVAR	TABLE 0550
	WRITE (12) T1AV1	TABLE 0560
	WRITE (12) T1AV2	TABLE 0570
C	RECORD IS FIRST CARD OF TVAR ARRAY	
	DO 36 I=1, 14	TABLE 0620
	K=1	TABLE 0630
29	READ (5, 901) (C(L), L=1, 6)	TABLE 0640
	ICARD=ICARD+1	TABLE 0650
	DO 34 L=1, 5, 2	TABLE 0660
	IF (C(L)+C(L+1)-199999.) 32, 30, 30	TABLE 0670
30	TVAR (I, L, K)=100000.	TABLE 0680
	TVAR (I, L, K)=100000.	TABLE 0690
	GO TO 4	TABLE 0700
32	TVAR (I, L, K)=C(L)	TABLE 0710
	TVAR (I, L, K)=C(L+1)	TABLE 0720
	IF (K-1) 33, 37, 37	TABLE 0730
33	K=K+1	TABLE 0740
34	CONTINUE	TABLE 0750
	GO TO 29	TABLE 0760
38	CONTINUE	TABLE 0770
36	WRITE (12) TVAR	TABLE 0780
	CSAVE=1.0	TABLE 0790
	READ (5, 901) (C(L), L=1, 6)	TABLE 0800
	ICARD=ICARD+1	TABLE 0810
	DO 30 I=1, 6	TABLE 0820
1036	CSAVE = CSAVE * C(I)	TABLE 0830
	IF (CSAVE) 37, 37, 37	TABLE 0840
37	CALL INTERP (6, 20, 1.0, P)	TABLE 0850
	CALL TIME	TABLE 0860
	CALL TIME	TABLE 0870
1037	CONTINUE	TABLE 0880
	REWIND 12	TABLE 0890
	REWIND 12	TABLE 0900
901	FORMAT (100, 7X, 19HERROR IN INPUT DATA, 126)	TABLE 0910
902	FORMAT (6512, 61)	TABLE 0920
927	FORMAT (H, 7X, 36HPRECEDING MAXIMUM TEMPERATURE INPUT,	TABLE 0930
	17HP01, =, 13)	TABLE 0940
	END	TABLE 0950
	SUBROUTINE INTERP (VAR, N, T, D)	
C	IF N=0, READ IN ALL TABLES, SET UP T ARRAY, AND EXIT.	
C	IF N=20 WRITE OUT ALL TABLES AND EXIT.	
C	IF N=1001, SET APPROPRIATE TABLES FROM TAPE,	
C	INTERPOLATE FOR VALUE OF VAR AND EXIT.	
C	IF N=5 THRU 13, INTERPOLATE FOR VALUE OF VAR AND EXIT.	
	DIMENSION TVAR (14, 2, 50), TVAR1 (35, 4, 35), TVAR2 (35, 4, 35)	
	1 READ (4, 15) H1(14)	
	COMMON TVAR1, TVAR2, TVAR	
	DO=0	
	IF (H) 1, 1, 10	
	K=1	
	REWIND 12	
	READ (12) TRAY	
	READ (12) TVAR1	
	READ (12) TVAR2	
C	T ARRAY HAS BEEN SET UP, NO. OF T RECORDS (K) IS SET, NOW	
C	PROCEED TO BRING IN PERMANENT ARRAYS.	

<pre> C READ (12) TVAR RETURN 10 IF (N-20) 100, 12, 11 11 WRITE (6, 911) CALL EXIT C WRITE OUT TABLES FOR N=20, T IS PAGE COUNT. 12 CALL ADATA LCT1=1 LPGCT=1 WRITE (6, 922) LPGCT LPGCT=LPGCT+1 DO 30 L=1, 14 13 NT=1 14 TEST1=TVAR(L, 1, NT)*TVAR(L, 2, NT) IF (TEST1-2.343446) 15, 15, 16 15 NT=NT+1 IF (NT-50) 14, 14, 1000 16 TVAR(L, 1, NT)=0.0 TVAR(L, 2, NT)=0.0 IF (NT-11515, 520, 515) C 40 INQUIRE THIS PAGE 520 WRITE (6, 9520) H(L) LCT1=LCT1+1 GO TO 510 515 NT=NT+1 WRITE (6, 916) H(L) L1=1 L2=AMINO(H, NT) LT=1 18 WRITE (6, 918) H(L), (TVAR(L, 2, LK), LK=L1, L2) IF (L2-NT) 19, 20, 20 19 L1=L1+1 L2=AMINO(L2+1, NT) LT=LT+1 GO TO 14 20 LCT1=LT+1+LCT1 530 IF (LCT1-2310, 21, 2) 21 WRITE (6, 922) LPGCT LPGCT=LPGCT+1 LCT1=1 30 CONTINUE PERIOD 12 DATA H(1)/48 CAG/ DATA H(2)/48 CBG/ DATA H(3)/48 WAG/ DATA H(4)/48 WBG/ DO 50 L=1, 4 IF (2*AY(L, 1)-99999.) 29, 29, 50 29 WRITE (6, 921) LPGCT 31 WRITE (6, 915) H(L) LPGCT=LPGCT+1 LCT1=1 DO 45 KK=1, 35 IF (2*AY(L, KK)-99999.) 32, 32, 45 32 WRITE (6, 916) TRAY(L, KK) LCT1=LCT1+1 NCT=1 34 TEST1=TVAR1(KK, L, NCT)*TVAR2(KK, L, NCT) IF (TEST1-2.343446) 35, 36, 36 35 NCT=NCT+1 IF (NCT-35) 34, 34, 1000 36 NCT=NCT-1 L1=1 L2=AMINO(R, NCT) 37 CONTINUE 38 DO 31 LT=L1, L2 IF (TVAR2(KK, L, LT)-100000.) 63, 60, 63 60 TVAR1(KK, L, LT)=0.0 TVAR2(KK, L, LT)=0.0 63 CONTINUE </pre>	<pre> INTR 1220 INTR 1230 INTR 1250 INTR 1260 INTR 1270 INTR 1280 INTR 1290 INTR 1290 INTR 1340 INTR 1350 INTR 1360 INTR 1370 INTR 1380 INTR 1390 INTR 1400 INTR 1410 INTR 1420 INTR 1430 INTR 1440 INTR 1450 INTR 1460 INTR 1470 INTR 1480 INTR 1490 INTR 1500 INTR 1510 INTR 1520 INTR 1530 INTR 1540 INTR 1550 INTR 1560 INTR 1570 INTR 1580 INTR 1590 INTR 1600 INTR 1610 INTR 1620 INTR 1630 INTR 1640 INTR 1650 INTR 1660 INTR 1670 INTR 1680 INTR 1690 INTR 1700 INTR 1710 INTR 1720 INTR 1730 INTR 1740 INTR 1750 INTR 1760 INTR 1770 INTR 1780 INTR 1790 INTR 1800 INTR 1810 INTR 1820 INTR 1830 INTR 1840 INTR 1850 INTR 1860 INTR 1870 INTR 1880 INTR 1890 INTR 1900 INTR 1910 INTR 1920 INTR 1930 INTR 1940 INTR 1950 INTR 1960 INTR 1970 INTR 1980 INTR 1990 INTR 2000 </pre>
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WRITE (6,938) H(L), (TVAR2(KK,L,LT),LT=L1,L2)
39 WRITE (6,939) (TVAR1(KK,L,LT),LT=L1,L2)
   LT=LCT+1
   IF (LCT-52) 66,65,65
65 WRITE (6,923) LPGCT
   LPGCT=LPGCT+1
   LCT=LCT+1
66 IF (L2-NCT) 40,41,41
40 L1=L1+8
   L2=L1+8 (L2+8,NCT)
   GO TO 38
41 IF (LCT-52) 45,42,42
42 WRITE (6,921) LPGCT
   LPGCT=LPGCT+1
   LCT=LCT+1
45 CONTINUE
50 CONTINUE
   IF (LBA-1) 1047,1046,1047
1046 N=0
1047 RETURN
C N=1 THRU 19 - INTERPOLATE ROUTINE
100 IF (N-4) 101,101,150
101 DO 104 I=1,35
   IF (TVAR1(N,I)-T) 104,105,105
104 CONTINUE
   GO TO 140
105 IF (I-1) 106,120,106
106 I=I-1
   TK=TVAR1(N,I)
   TK1=TVAR1(N,I+1)
   DO 107 J=1,35
   IF (TVAR1(IM1,N,J1)-P) 107,107,108
107 CONTINUE
   GO TO 120
108 IF (J1-1) 109,120,109
109 IF (TVAR1(I,N,J1)+TVAR2(IM1,N,J1)-2.0*99999.1) 110,120,120
110 DIFF1=TVAR1(IM1,N,J1)-TVAR1(I,N,J1)
   DIFF2=TVAR1(I,N,J1)-TVAR1(I+1,N,J1)-D
   PC1=(DIFF1/DIFF2)*(TVAR2(IM1,N,J1)-TVAR2(IM1,N,J1+1))+
   TVAR2(I,N,J1)
   DO 111 J2=1,35
   IF (TVAR1(I,N,J2)-P) 111,111,112
111 CONTINUE
   GO TO 120
112 IF (J2-1) 113,120,113
113 J2=J2-1
   PC2=TVAR1(I,N,J2)+TVAR2(I,N,J2)-2.0*99999.1) 114,120,120
114 DIFF=TVAR1(I,N,J2)-TVAR1(I,N,J2)
   DIFF1=TVAR1(I,N,J2)-TVAR1(I,N,J2+1)-D
   PC1=(DIFF1/DIFF2)*(TVAR2(I,N,J2)-TVAR2(I,N,J2+1))+TVAR2(I,N,J2)
   PC2=(DIFF1/TK1)/(TK-TK1)*(PC2-PC1)+PC1
110 PE=JN4
120 N=0
   IN=1
   GO TO 12
150 J=N-4
   IF (N-151) 155,1000,1000
155 CONTINUE
   DO 151 I=1,50
   IF (TVAR(N,1,I)-T) 151,151,152
151 CONTINUE
   GO TO 120
152 IF (TVAR(N,1,I)+TVAR(N,2,I)-2.0*99999.1) 153,120,120
153 IF (I-1) 154,120,154
154 IF (I-1) 154,120,154
   VAR=(TVAR(N,1,IP)-TVAR(N,1,IP'))/(TVAR(N,2,IP)-TVAR(N,2,IP'))
   GO TO 120
1000 CONTINUE
   CALL OUTPUT

```

AD-A125 730

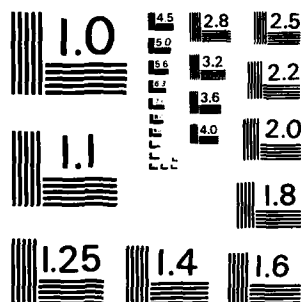
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MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

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CALL DUMP
RETURN
911 FORMAT (1H0,7X,13HILLEGAL CALLING SEQUENCE TO INTRP)
916 FORMAT (1H0,6X,11HINPUT TABLE,2X,A5,/)
918 FORMAT (1H0,A5,1PHE15.6)
919 FORMAT (1H,2X,1PHE15.6)
920 FORMAT (1H1,2X,120X,4PAG,1X,1)
914 FORMAT (1H1,4X,14HTEMPERATURE = ,F12.4,/)
918 FORMAT (1H0,2X,1PHE15.6)
919 FORMAT (1H,2X,1PHE15.6)
9520 FORMAT (1H0,7X,14HNO INPUT FOR TABLE, A6)
END
C.....
SUBROUTINE DUMP
CALL EXIT
RETURN
END
C.....
SUBROUTINE ADATA
DIMENSION HT(18),H(14),HT1(19),H1(14)
COMMON HT,H
DATA HT1/4H CAG,4H COG,4H KAG,4H KAG,4H CAL,
1 4H COL,4H LA,4H LB,4H PBO,4HROAL,
2 4HROAL,4H KAG,4H KAG,4HVIAG,4HVIAG,
3 4HVIAG,4HVIAG,4HVIAG,4HVIAG,
DATA H1/4H CAL,4H CBL,4H HLA,4H HLB,4H PBO,
1 4HROAL,4HROBL,4H KAG,4H KAG,4HVIAG,
2 4HVIAG,4HVIAG,4HVIAG,4HVIAG,
DO 10 I=1,18
HT(I)=HT1(I)
10 CONTINUE
DO 20 I=1,14
H(I)=H1(I)
20 CONTINUE
RETURN
END

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