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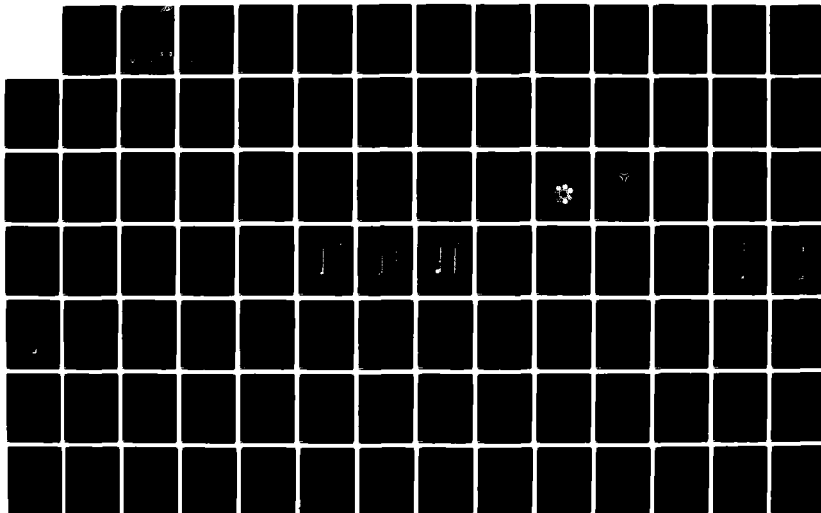
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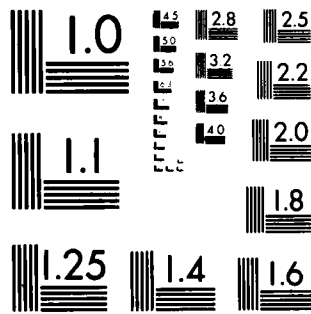
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Part II. Parametric Analysis and Model City Simulations

By
D. A. Larson
R. D. Small

November 1982

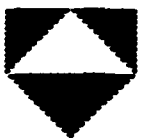
Final Report
Contract EMW-C-0747, Work Unit 2564E

For
Federal Emergency Management Agency
National Preparedness Programs
Washington, D.C. 20472

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BLOCK 20 (cont.)

burning rate is increased. A sample calculation illustrates the change in fire-wind velocities as the fire evolves over time.

Calculations of the burning region for three model urban areas show the influence of building density and urban sprawl on the resulting fire environment. An additional set of predictions accounts for reduction of the fire intensity by blast in the urban center. For the latter cases, the temperature distribution is changed markedly, though the magnitude of the induced fire winds is not appreciably reduced.

PREFACE

This report is the second of the two-part documentation of Pacific-Sierra Research Corporation's analysis of the large urban fire environment. This part presents calculations of the characteristics of nuclear-weapon-ignited fires. Part I develops the theory underlying the analysis. All work was performed for the Federal Emergency Management Agency under contract EMW-C-0747; the technical monitor was Dr. David Bensen.

The authors gratefully acknowledge the contributions of Dr. Harold L. Brode.

SUMMARY

This report considers the large-fire environment that would occur in an urban area subject to a nuclear weapon explosion. The effects of system parameters are explored in a sensitivity study, and results for three model cities are presented.

The examples are characterized by extensive areas simultaneously burning, strong buoyancy, and large temperature gradients. Several such fires occurred during World War II. Though those fires were dramatic in intensity and destructiveness, each involved a relatively small area. A nuclear weapon explosion could generate a far larger area fire and a more severe fire environment. This report is intended to define such large fires. The results should be applicable for damage evaluation, formulation of shelter requirements, and rescue planning.

The calculations are based on the theory developed in Part I of this report, which is applicable to the fire zone and the volume immediately above it (turning region). The effects of variable area heating, turbulence, strong buoyancy, large temperature changes, and radiation are treated. The induced fire winds and rapid temperature changes at the fire periphery are uniquely determined by the use of jump conditions. Simulations of the Hamburg firestorm and a large Flambeau fire agreed well with available data.

The parametric analysis considers a large area fire and the effects of fire size, heating rates, mixing coefficients, and hot gas/smoke radiation. The results show the influence of those variables on the induced fire winds, mean temperature, and pressure gradients. In general, an increase in either the fire size or heating rate raises the mean temperature levels and the induced fire-wind velocities. For the larger heat release rates or fire sizes, the attendant increases in mean temperature and velocity are limited by compressibility effects.

Fires such as may result from a megaton-yield explosion are analyzed for three model urban areas. Each city is characterized by

a high-density center, a surrounding belt of mixed residential/ industrial construction, and a lower density suburban belt. Each model city portrays a different degree of building density and urban sprawl. The results illustrate how a particular city geometry affects the velocity and temperature fields for a given fire. An additional series of computations considers reduction of the fire area by severe blast damage and debris formation. For those calculations, complete burning was allowed in an annular area, with the fire intensity significantly reduced in the center.

Finally, the model is employed to estimate the behavior of the velocity and temperature fields as a function of fire evolution. Those calculations may indicate the most appropriate periods for effecting rescue operations as well as provide an estimate of time-dependent shelter loadings.

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SYMBOLS

MATHEMATICAL SYMBOLS

- A = dimensionless constant, gH/U^2
- c_p = specific heat capacity at constant pressure
- E_B = average heat released per unit weight of all combustibles
- ξ_1 = effective kinematic viscosity (for turbulent flow)
- f_{Bu} = building density (ratio of area covered by buildings to total area)
- F = dimensionless measure of QH [defined by Eq. (13)]
- g = gravitational acceleration
- H = maximum height of flames
- k^* = reciprocal of graybody radiation mean free path
- k_1 = effective thermal conductivity (for turbulent flow)
- K_1 = dimensionless heat-diffusion coefficient
- L_s = average fuel loading per building story
- L_T = average areal fuel loading
- M_1 = dimensionless momentum-diffusion coefficient
- N_s = average number of building stories
- P = perturbation pressure (dimensionless)
- P_a = ground-level atmospheric pressure in far field
- P_{max} = maximum perturbation pressure
- q = dimensionless spatial distribution of volumetric heat-addition rate
- Q = volumetric heat-addition-rate scale
- Q_A = areal heat-addition rate

- r = radial position coordinate (dimensionless, except in Figs. 24 through 26)
- R = fire radius
- T = temperature (dimensionless)
- T_a = ground-level atmospheric temperature in far field
- T_{\max} = maximum temperature
- u = radial velocity (dimensionless)
- u_{\max} = maximum radial velocity
- U = radial velocity scale
- v = vertical velocity (dimensionless)
- v_{\max} = maximum vertical velocity
- y = vertical position coordinate (dimensionless)
- γ = specific heat ratio
- δ = dimensionless constant, $U^2 / (P_a / \rho_a)$
- ϕ = radial dependence of model-city heat release rates in blast-modified cases
- ρ = density (dimensionless)
- ρ_a = ground-level atmospheric density in far field
- σ = radiation coefficient (dimensionless constant), $4\pi\hat{\sigma}T_a^4k^*/Q$
- $\hat{\sigma}$ = Stefan's constant

SYMBOLS[†] USED IN DOCUMENTATION OF PREDICTION ALGORITHM

- E_j = intermediate variable [defined by Eq. (A.37)]
- $\vec{F}, \vec{G}, \vec{H}$ = vectors of functions that are to be driven to zero (by proper choice of \vec{x}) in Newton iterations, and that represent discretized model equations on individual lines of constant y

[†]Mathematical variables only--not FORTRAN labels.

- G_i, H_i = i th components of \vec{G} and \vec{H}
- i = radial position index ($1 \leq i \leq M + 1$)
- j = vertical position index ($1 \leq j \leq N + 1$)
- J = $\partial \vec{F} / \partial \vec{x}$ Jacobian matrix
- k = vector component index ($1 \leq k \leq 2M$)
- ℓ = index for $\alpha, \beta, \gamma, \delta$ Jacobian elements ($1 \leq \ell \leq M$)
- m, n = iteration counters in Newton and shooting iterations
- M, N = number of finite difference cells in radial and vertical directions, respectively (see footnote, p. 68)
- p = index for $\alpha, \beta, \gamma, \delta$ Jacobian elements ($0 \leq p \leq 2$)
- $P_{i,j}$ = $P(r_i, y_j)$
- P_{old}, P_{new} = successive discretized forms of P in shooting iterations
- $(Qq)_{max}$ = maximum value of Qq
- r_i = $(i - 1) \Delta r$
- R_i = sum of all terms in G_i , for any line $y = y_j$, j fixed, that depend only on $y = y_{j-1}$ data
- t = time
- \vec{T} = vector of discrete T values for line of constant y
- $T_{i,j}$ = $T(r_i, y_j)$
- T_{old}, T_{new} = successive discretized forms of T in shooting iterations
- \vec{u} = vector of discrete u values for line of constant y
- $u_{i,j}$ = $u(r_i, y_j)$
- u_{old}, u_{new} = successive discretized forms of u in shooting iterations
- $v_{i,j}$ = $v(r_i, y_j)$
- w = generic variable
- $w_{i,j}$ = generic finite-difference variable

\vec{x} = vector of discrete u and T values for line of constant y

\tilde{x}_k = k th component of \vec{x}

$\vec{x}_{old}, \vec{x}_{new}$ = successive discretized forms of \vec{x} in Newton iterations

$y_j = (j - 1) \Delta y$ for $j \leq 5M + 1$; $5 + (j - 5M - 1)(2 \Delta r)$ for $j > 5M + 1$

y_{max} = maximum value of y considered

$\alpha_{p,l}$ = nonzero element in $\partial \vec{G} / \partial \vec{u}$ Jacobian ($0 \leq p \leq 2, 1 \leq l \leq M$)

$\beta_{p,l}$ = nonzero element in $\partial \vec{G} / \partial \vec{T}$ Jacobian ($0 \leq p \leq 2, 1 \leq l \leq M$)

$\gamma_{p,l}$ = nonzero element in $\partial \vec{H} / \partial \vec{u}$ Jacobian ($0 \leq p \leq 2, 1 \leq l \leq M$)

$\vec{\delta}$ = vector difference between successive values of \vec{x} in Newton iterations

δ_k = k th component of $\vec{\delta}$

$\delta_{p,l}$ = nonzero element in $\partial \vec{H} / \partial \vec{T}$ Jacobian ($0 \leq p \leq 2, 1 \leq l \leq M$)

$\Delta r, \Delta y$ = cell width and height, respectively

Π_i = sum of all terms in H_i , for any line $y = y_j$, j fixed, that depend only on $y = y_{j-1}$ data

$\rho_{i,j} = \rho(r_i, y_j)$

ω = relaxation coefficient in shooting iterations

I. INTRODUCTION

This report presents predictions of the temperatures, pressures, and high-speed winds created by large urban fires. The dependence of those quantities on fire size, burning rate, and various other parameters is explored, and fires in model U.S. cities are examined. Simulations in which nuclear-weapon-ignited fires are extinguished in the center by blast are compared with those in which the fires continue to burn. The model developed in Part I of this report was used to calculate all values. That model may be readily extended to obtain estimates of oxygen depletion and noxious gas buildup as well and, hence, a fairly complete baseline description of the environment facing civil defense personnel when large urban areas are burning.

Most previous research concerning the hydrothermodynamics of free-burning fires has been restricted to the weakly buoyant free-convection plume. The resulting theories describe the basic flow in the middle and upper parts of a long, thin plume over a small fire, and may be relevant to some portion of the convection column generated over a large urban fire [e.g., Morton, Taylor, and Turner, 1956; Murgai and Emmons, 1960; Yokoi, 1960; Nielsen and Tao, 1965]. However, as discussed in Part I, such theories are inapplicable in and around the combustion zone, so they cannot be used to predict the environment and high-speed surface winds induced by large fires.

Smith, Morton, and Leslie [1975] present global computations of the hydrothermodynamic environment near the combustion zone as well as above it. In their calculations, all components of the fire-generated flow field--e.g., surface inflow, the convection column, the far field--are considered collectively, and the strong coupling between dynamic forces and the induced surface winds is demonstrated. In and around the combustion zone, however, density changes are assumed to be small (Boussinesq approximation), heat is input at the boundary rather than in volume, and not much resolution is sought.

Further, the computations are not specifically designed for the analysis of large urban fires.

In a more recent study, Brode, Larson, and Small [1982] adopt the basic computational approach of Smith, Morton, and Leslie in studying the mesoscale motions generated by large fires, but eliminate some of the stated deficiencies. Their work is directly aimed at the definition of the large urban fire environment, and arbitrary changes in temperature and density are allowed in the near-fire region. The resolution in that region is still rather coarse, however.

The component analysis introduced by Small, Larson, and Brode [1981] provides an alternative to the global-calculation analysis in which fine resolution near the fire can be obtained economically. Using that approach, the components of the flow field are analyzed separately, and then combined by means of appropriate matching conditions. The special features of each component are thus considered in greater detail, and resolution is enhanced.

The key element in the component analysis is the description of the environment in and around the fire, since the entire flow is driven by the hydrothermodynamic interactions in that region. The turning-region model developed in Part I provides the key description. That model decouples from those for the other flow components, and thus can be used independently to make detailed predictions of the near-fire environment. Such a capability extends our understanding of fire dynamics and is the basis for the present analysis.

We use the turning-region model to generate the large urban fire predictions. In that model, a volume heat source represents the net effect of the combustion process, and the induced flow is taken to be axisymmetric and quasi-steady. The flow is also assumed compressible, to permit arbitrary changes in temperature and density. A one-parameter eddy-viscosity model is used to describe the turbulent transfer of heat and momentum, and a graybody approximation to model hot gas and smoke radiation. Finally, jump conditions are derived to account for the rapid changes in physical quantities at the fire periphery. They effect model problem closure, and decouple the turning-region analysis from that of other component flows. An iterative

finite-difference scheme, documented in the Appendix, is employed to solve the resultant boundary value problem.

We study the large urban fire environment in two ways--by varying parameters and by simulating three model cities. In the parametric analysis, excursions are made about a baseline in which the fire radius is taken to be 10 km and the characteristic heat release rate is consistent with that of the 1943 Hamburg firestorm [DCPA, 1973]. Predictions of hurricane-force winds are typical in the parameter excursions as well as the model city simulations, which consider fires of radius 12 km with spatially varying heating rates on the order of those in the parametric analysis. That is believed to be about the size of the fire that would result from a near-surface nuclear burst of about 1 Mt [Johnson and Larson, 1982].

Section II presents the results of the parametric analysis. As expected, temperatures and fire-wind velocities increase with an increase in either fire size or burning rate, though the fire-wind variations are not as rapid as the linear scaling law of Part I suggests. Also as expected, temperatures and winds increase with a decrease in radiation intensity; winds also increase with a decrease in the magnitude of the eddy coefficient for turbulent momentum transfer. The temperature field is relatively insensitive to changes in that coefficient, however, and both winds and temperature are relatively insensitive to changes in the corresponding coefficient for turbulent heat transfer.

The basic flow pattern also seems to be relatively insensitive to variations in the specific dependence of the heat release rate on spatial position, though such variations modify velocities slightly and may significantly change the temperature field. That behavior is found when model predictions for a uniformly heated, circular fire are compared with those for an annular fire. The comparison indicates how partial blast extinguishment would modify the hydrothermodynamic environment generated by a nuclear-weapon-ignited fire. It also provides a first look at the catastrophic effect of multiple weapon bursts.

Section III describes the model city simulations. Three model cities are defined: one that is lightly built-up, intended to represent new, sprawling cities; one that is heavily built-up, intended to represent old, congested cities; and one of intermediate building density. For each city, spatially dependent fuel loadings and peak-period burning rates for a baseline fire are estimated, and the resulting temperatures, pressures, and fire winds predicted. Those calculations are repeated for a scenario in which the fire is ignited by a nuclear burst and the city partially destroyed by the attendant blast.

Somewhat surprisingly, despite a significant difference in model-city temperature predictions for the "blast" and "no blast" simulations, the corresponding difference in induced fire winds is fairly small. Such behavior suggests that the winds and wind damage resulting from nuclear-weapon-ignited fires may be relatively independent of the degree of blast extinguishment--unless, of course, extinguishment is nearly complete.

As expected, however, temperatures and fire-wind velocities all increase with building density. For a given fire size, the large-fire threat facing civil defense workers will thus be more severe in the tall, heavily loaded cities than in the short, lightly loaded cities. In general, however, the shorter cities sprawl out over greater land areas than do taller ones of comparable population, and are thus capable of supporting larger fires. The large-fire threat for any given population size may thus be greatest in some of the shorter cities, especially if the fires result from multiple nuclear bursts. Comparisons between specific cities must be made individually.

Finally, Sec. IV presents a sample application of the quasi-steady model of Part I to obtain time-dependent predictions. The sample case considers multiple-fuel-bed Flambeau fires, the largest area fires for which a reasonable body of technical data exists [Countryman, 1969; Palmer, 1981].

II. PARAMETRIC ANALYSIS OF LARGE-FIRE ENVIRONMENT

Using the turning-region model of Part I, this section explores the dependence of the large-fire environment on fire size (height and radius), burning rate (spatial average and form), degree of turbulent mixing, and degree of hot-gas radiation.

Briefly, with r , y , u , v , ρ , T , and P denoting dimensionless radial and vertical position, radial and vertical velocity, density, temperature, and perturbation pressure, respectively, the turning-region equations are as follows:

$$\frac{\partial}{\partial r} (r\rho u) + \frac{\partial}{\partial y} (r\rho v) = 0 , \quad (1a)$$

$$\rho \left(u \frac{\partial u}{\partial r} + v \frac{\partial u}{\partial y} \right) = - \frac{\partial P}{\partial r} + M_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u}{\partial r} \right) - \frac{u}{r^2} \right) , \quad (1b)$$

$$\frac{\partial P}{\partial y} + A\rho = 0 , \quad (1c)$$

$$\rho \left(u \frac{\partial T}{\partial r} + v \frac{\partial T}{\partial y} \right) = q(r, y) - \sigma(T^4 - 1) + K_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) \right) , \quad (1d)$$

$$\rho T = 1 , \quad (1e)$$

where the dimensionless parameters are

$$A = \frac{gH}{U^2} ,$$

$$U = \frac{\gamma - 1}{\gamma} \left(\frac{R}{H} \right) \left(\frac{QH}{P_a} \right) ,$$

$$\sigma = 4\pi\hat{\sigma}T_a^4 \left(\frac{k^*H}{QH} \right) , \quad (2)$$

and

$$M_1 = \frac{\mathcal{E}_1 / \rho_a}{UR} ,$$
$$K_1 = \frac{k_1 / c_p \rho_a}{UR} . \quad (3)$$

Here, R and H denote fire radius and maximum flame height; Q the scale and $q(r, y)$ the spatial distribution for the rate of volumetric heat addition; and P_a , ρ_a , and T_a ground-level atmospheric pressure, density, and temperature in the far field, respectively. Further, $\hat{\sigma}$ is Stefan's constant, k^* is the reciprocal of the graybody radiation mean free path (assumed constant), and \mathcal{E}_1 and k_1 are dimensional eddy coefficients for turbulent momentum and heat transfer.[†]

We consider the dependence of model predictions on the size parameters R and H, the burning rate parameters QH^{++} and $q(r, y)$, the radiation mean free path k^{*-1} , and the dimensionless eddy coefficients M_1 and K_1 . The terms \mathcal{E}_1 / ρ_a and $k_1 / c_p \rho_a$ remain constant, except when variations with M_1 and K_1 are specifically studied.

BASELINE ANALYSIS

As a baseline case, we consider an axisymmetric fire with the following parameters:

$$R = 10 \quad \text{km} , \quad (4a)$$

$$H = 100 \quad \text{m} , \quad (4b)$$

$$QH = 5.7 \times 10^4 \quad \text{cal/m}^2\text{-sec} , \quad (4c)$$

[†]As in Part I, g is the acceleration due to gravity, γ the specific heat ratio, and U the radial velocity scale. The vertical velocity is scaled with $(H/R)U$, temperature and density with T_a and ρ_a , and perturbation pressure with δP_a , where $\delta = U^2 / (P_a / \rho_a)$.

⁺⁺QH is used as a parameter instead of Q because areal heat release rates are encountered much more frequently than volumetric rates in the large-fire literature.

$$q(r, y) = \begin{cases} 1.6 & \text{for } 0 \leq y \leq 0.25 \\ 1.6 \left(\frac{4}{3}\right) (1 - y) & \text{for } 0.25 \leq y \leq 1.0 , \\ 0 & \text{for } y \geq 1.0 \end{cases} \quad (4d)$$

$$M_1 = K_1 = 0.2 , \quad (4e)$$

$$k^{*-1} = 20 \quad \text{m} . \quad (4f)$$

The heat release rate in this case is the same as that in our simulation in Part I of the 1943 Hamburg firestorm, and is characteristic of the rates estimated in Sec. III for a variety of U.S. cities. The parameter QH is the nominal specific scale rate suggested by DCPA [1973]. The form of $q(r, y)$ represents maximum heating in and around the fuel zone and a decrease in heating with increased altitude. All other parameter choices used in Eqs. (4) are of the same order as those used in the model city analysis.

From Eqs. (2) and (4), the radial velocity scale for the baseline case is

$$U = 67.2 \quad \text{m/sec} , \quad (5)$$

and

$$\begin{aligned} A &= 0.217 , \\ \sigma &= 0.110 . \end{aligned} \quad (6)$$

The temperatures, pressures, and velocities predicted for this case are summarized in the vector, contour, and profile plots of Figs. 1 through 5. The flow field is illustrated in Fig. 1. Temperature and pressure contours are plotted in Figs. 2 and 3, respectively. The temperature attains a maximum in the fire center at the top of the maximum heating zone ($y = 0.25$). It then decreases rapidly with increased altitude, the flow becoming weakly buoyant above three

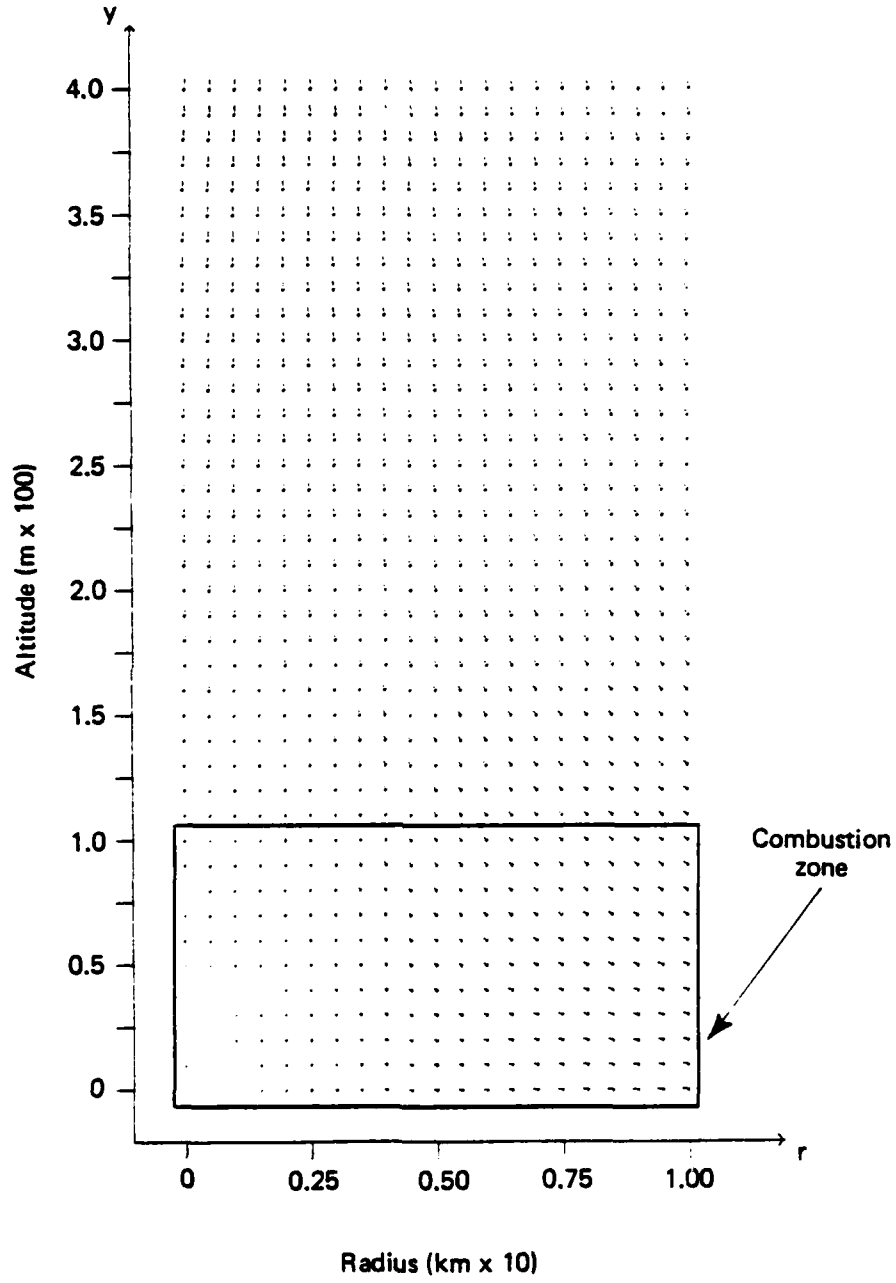


Fig. 1--Velocity field for baseline fire

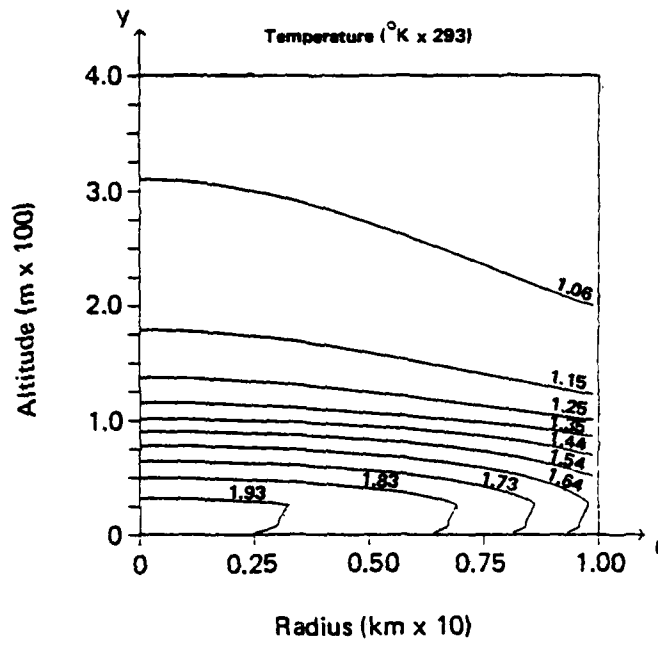
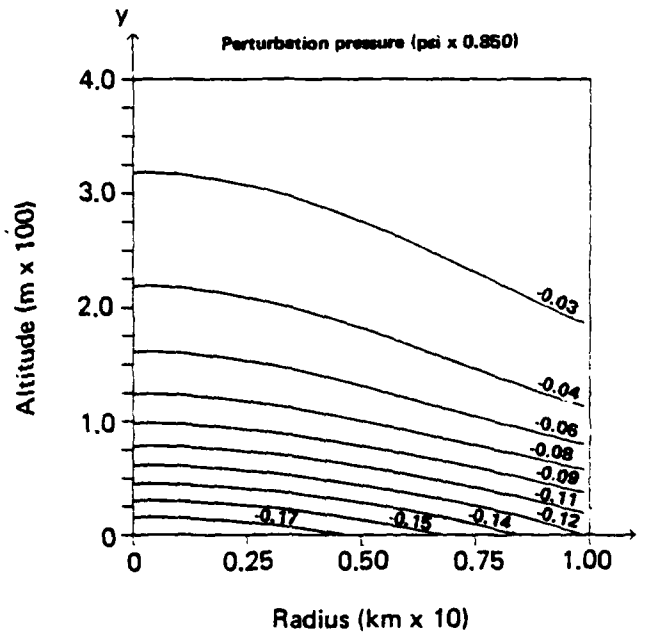


Fig. 2--Temperature contours for baseline fire



Note: Perturbation pressure is $\delta P_0 (P + Ay)$; see footnote † on p. 6.

Fig. 3--Pressure contours for baseline fire

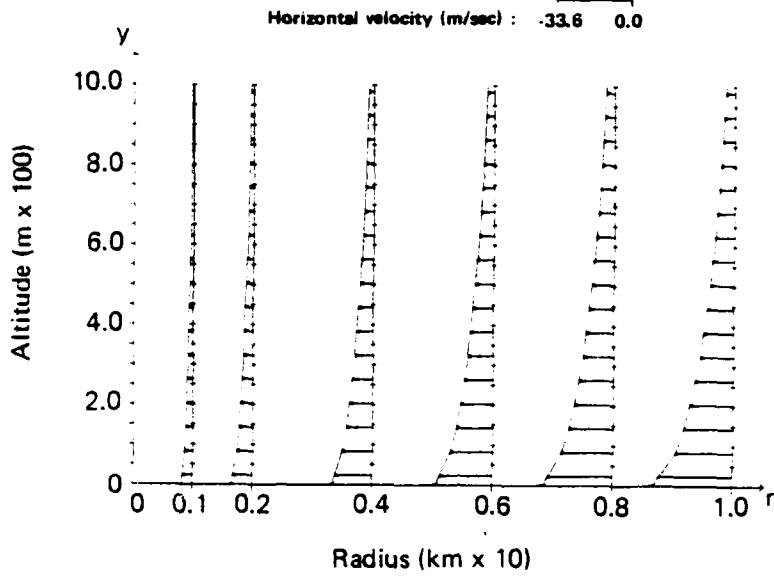


Fig. 4--Radial velocity profiles for baseline fire

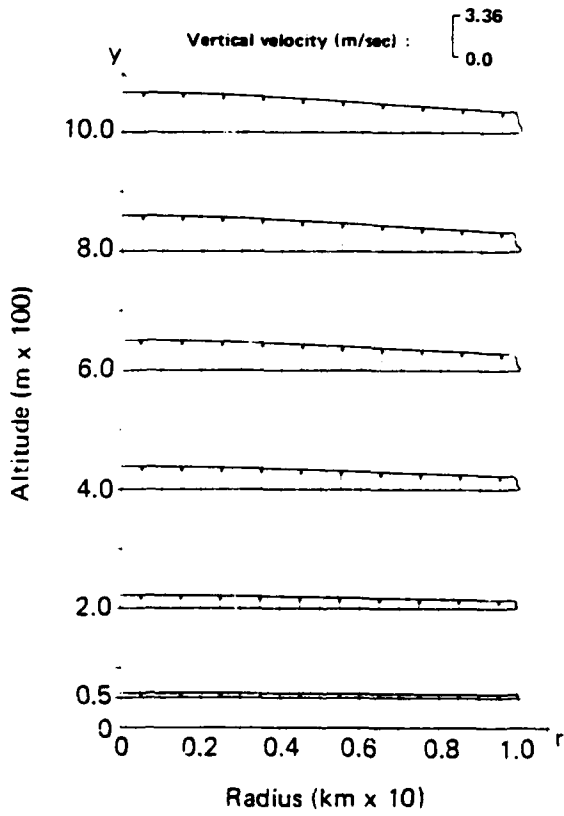


Fig. 5--Vertical velocity profiles for baseline fire

flame heights. As expected, the pressure drop is maximum at ground level. Profiles of the horizontal inflow induced by the pressure drop are presented in Fig. 4, and those of the resulting upflow in Fig. 5. The maximum inflow occurs on the ground at the fire periphery. The upflow profiles are nearly "top-hat."

In most parameter excursions, the hydrothermodynamic predictions are qualitatively the same as those for the baseline case. The results are summarized below.

DEPENDENCE ON FIRE SIZE AND BURNING RATE SCALE

We consider first the dependence of the large-fire environment on R, H, and QH--the parameters that collectively describe the fire. For nuclear-weapon-ignited fires, the effective value of R will depend on the number and yields of bursts, ambient atmospheric conditions, various characteristics of the preblast urban area (e.g., flammability of exposed combustibles, susceptibility of structures to secondary fires), and a variety of lesser factors [Johnson and Larson, 1982]). Parameters QH and H will depend on the type and distribution of combustibles in the postblast environment, as detailed in Sec. III.

Tables 1 through 3 summarize the variations that we consider in fire size and burning rate, and the corresponding changes in model parameters that follow from Eqs. (2) and (3). In most excursions, the variations are of the type in Table 2, where the parameter under study is simply doubled or halved. The additional variations in Tables 1 and 3 are considered in order to refine the excursions that we believe to be of greatest intrinsic interest--i.e., those defining the dependence of fire winds and temperatures on fire radius (weapon coverage) and intensity (concentration of combustibles).

Figures 6 through 11 summarize the results of the fire size and burning rate excursions. The dependence of fire winds and temperatures on fire size and intensity is plotted in Figs. 6 through 8. As expected, the maximum induced velocity u_{\max} and the maximum temperature T_{\max} both increase with either radius or intensity. The increases are nearly linear for relatively small radii and heating rates,

Table 1

PARAMETER DEPENDENCE ON FIRE RADIUS

R (km)	U (m/sec)	A	σ	M_1, K_1
2.5	16.8	3.4720	0.110	3.200
5.0	33.6	0.8680	0.110	0.800
10.0	67.2	0.2170	0.110	0.200
15.0	100.8	0.0964	0.110	0.089
20.0	134.4	0.0534	0.110	0.050

Table 2

PARAMETER DEPENDENCE ON FIRE HEIGHT

H (m)	U (m/sec)	A	σ	M_1, K_1
50	134.4	0.0272	0.055	0.4
100	67.2	0.2170	0.110	0.2
200	33.6	1.7360	0.220	0.1

Table 3

PARAMETER DEPENDENCE ON BURNING RATE SCALE

QH (kcal/m ² -sec)	U (m/sec)	A	σ	M_1, K_1
14.35	16.8	3.4720	0.4400	0.80
28.70	33.6	0.8680	0.2200	0.40
57.40	67.2	0.2170	0.1100	0.20
86.10	100.8	0.0964	0.0825	0.15
114.80	134.4	0.0534	0.0550	0.10

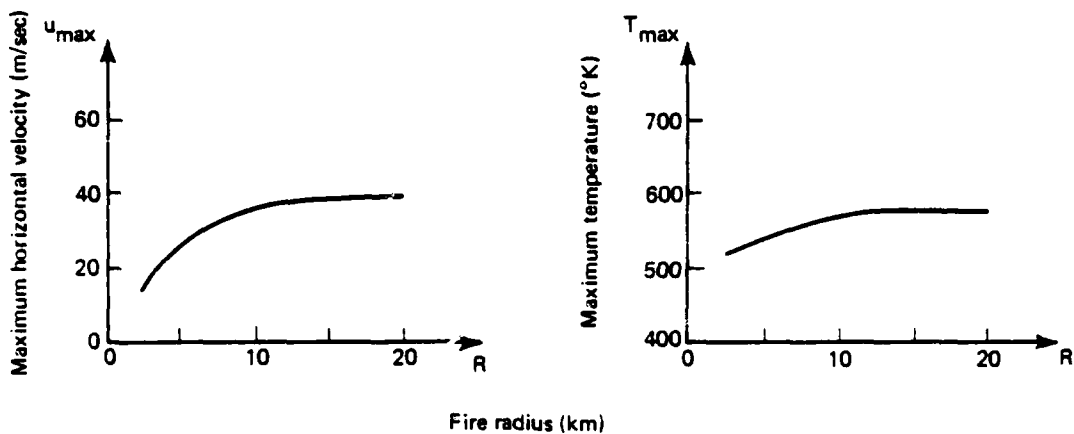


Fig. 6--Dependence of maximum radial velocity and temperature on fire radius

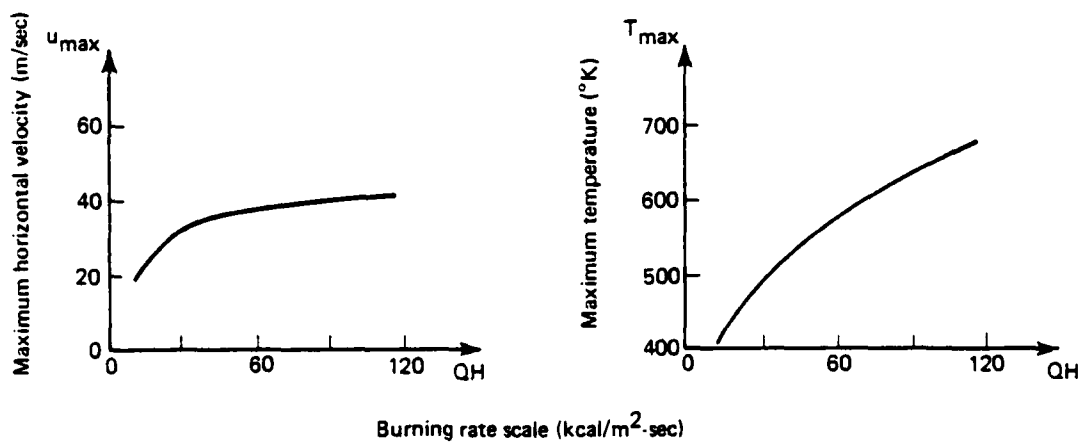


Fig. 7--Dependence of maximum radial velocity and temperature on burning rate scale

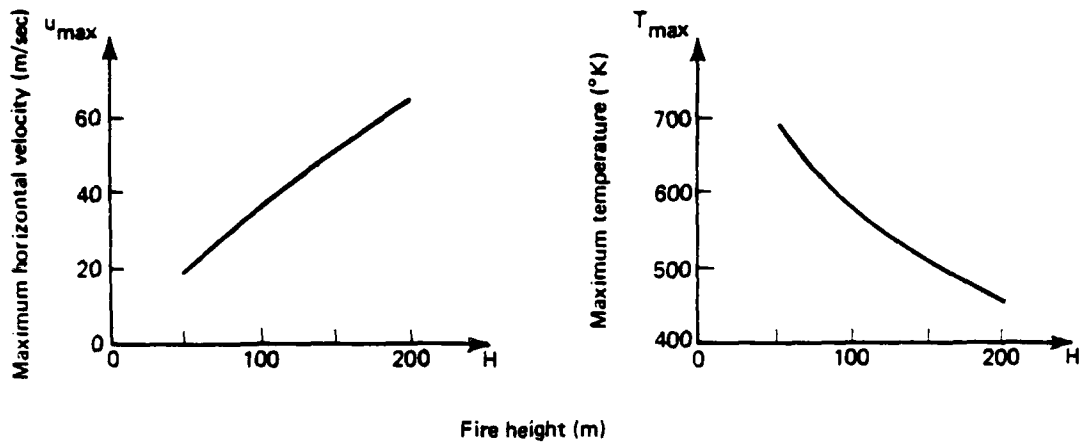


Fig. 8--Dependence of maximum radial velocity and temperature on fire height

but tail off markedly for fires of much greater size and intensity. In fact, u_{\max} and T_{\max} appear to asymptote toward finite values as R or $QH \rightarrow \infty$. Surprisingly, the apparent asymptotic values of u_{\max} as either R or $QH \rightarrow \infty$ are almost identical. T_{\max} , however, exhibits a stronger dependence on QH than on R .

Figure 8 describes the variation of u_{\max} and T_{\max} with fire height. As H increases, u_{\max} increases but T_{\max} decreases. Such behavior is presumably explained as follows. With QH fixed, Q decreases when H increases. Temperature decreases correspondingly, because of the shorter characteristic residence times of fluid particles in the heating zone. Lower temperatures might be expected to decrease velocities as well. However, a smaller fraction of the constant total heat release QH is radiated away at lower temperatures, so higher kinetic energies can be supported.

The results in Fig. 8 must be interpreted with some care. Whenever H is varied, QH may be varied as well. For example, when buildings have the same construction and contents, both H and QH should

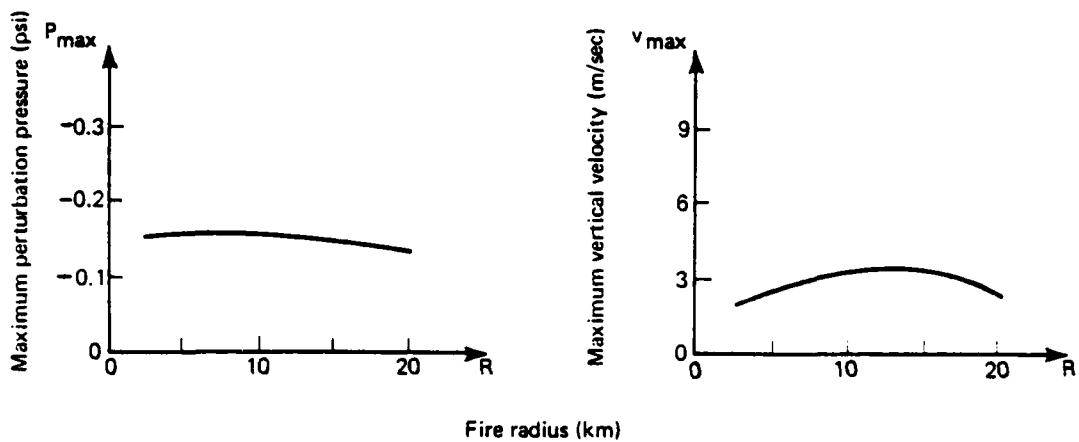


Fig. 9--Dependence of maximum perturbation pressure and vertical velocity on fire radius

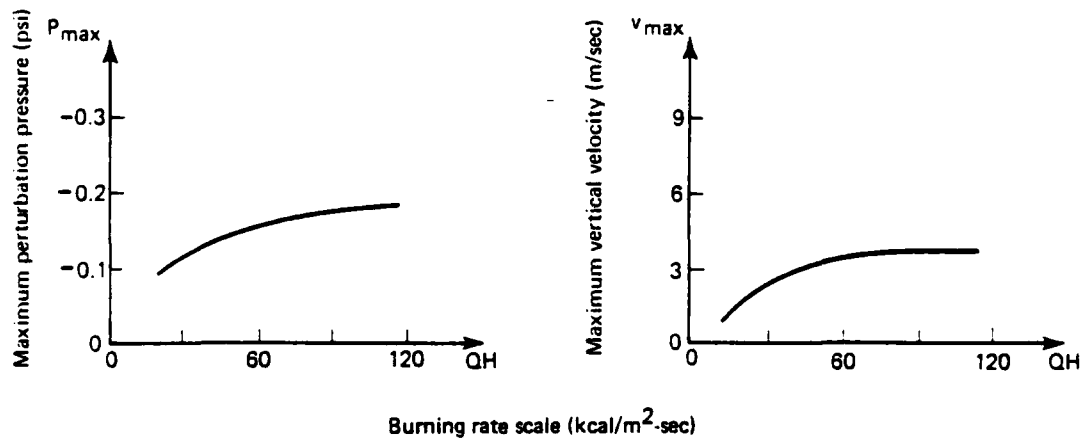


Fig. 10--Dependence of maximum perturbation pressure and vertical velocity on burning rate scale

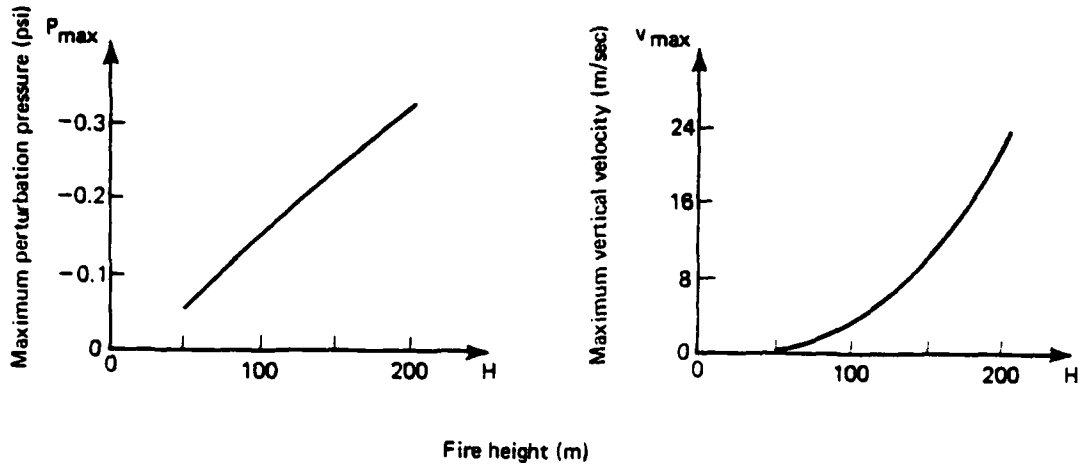


Fig. 11--Dependence of maximum perturbation pressure and vertical velocity on fire height

increase with the number of stories. In such cases, the results in Fig. 7 may be more applicable.

Figures 9 through 11 plot the dependence of maximum pressure drop P_{\max} and vertical velocity v_{\max} on R , QH , and H . As expected, vertical velocities generally increase with radial velocities. Greater pressure gradients foster greater radial velocities, and are less strongly correlated with temperature variations than Eq. (1c) suggests.

DEPENDENCE ON BURNING RATE SPATIAL DISTRIBUTION

We now briefly explore the effects of different spatial dependences of the heat release rate $q(r, y)$. We consider several sinusoidal and annular variations in radial dependence, but just one in vertical dependence. Further variations are considered in the model city analysis in Sec. III.

Table 4 summarizes the variations in $q(r, y)$ and the resulting hydrothermodynamic changes. As expected, a relatively high-frequency

Table 4

VELOCITY, TEMPERATURE, AND PERTURBATION PRESSURE MAXIMA
FOR VARIOUS BURNING RATES

Type of Fire	$\left(\frac{q(r, y)}{\{q(r, y)\}_{\text{baseline}}} \right)$	u_{max} (m/sec)	T_{max} (°K)	P_{max} (psi)	v_{max} (m/sec)
Baseline	1.0	37.2	570	0.156	3.26
Sinusoidal					
Low-frequency	$1.0 - \cos 4\pi r$	38.1	588	0.160	3.50
High-frequency	$1.0 - \cos 20\pi r$	37.2	573	0.157	3.28
Annular	$\left\{ \begin{array}{l} 0.0 \text{ for } 0 \leq r < 0.5 \\ 1.0 \text{ for } 0.5 \leq r \leq 1 \end{array} \right\}$	33.9	515	0.119	2.46

radial variation on the baseline heat-release distribution has little effect. Surprisingly, however, a fairly low-frequency radial variation also produces trivial changes in velocity, temperature, and perturbation pressure. Those results suggest that the large-fire environment may be more sensitive to the basic scale for heat release than to the specific spatial distribution of the heat release rate (see Fig. 7).

That conclusion is further supported by a comparison of the results in Table 4 for the annular fire with those for the circular baseline fire. The ground area covered by the annular fire--and hence its total heat release--is 25 percent less than that of the baseline fire. The corresponding drop in maximum temperature difference from ambient (i.e., in $T_{\text{max}} - T_a$) is just slightly less--20 percent. The drop in u_{max} is significantly less--only 9 percent. Such behavior suggests that the maximum winds and temperatures generated by nuclear-weapon-ignited urban fires may be relatively insensitive to changes in the geometry and loading of the blast-damaged region, provided of course that the blast does not nearly extinguish the fire.

Figures 12 and 13 compare the temperature and flow fields of the baseline and annular fires. Since the fire is located only in the annulus, the highest temperatures occur near the inner edge of the fire annulus ($r = 0.5$), rather than on the symmetry axis. The resulting

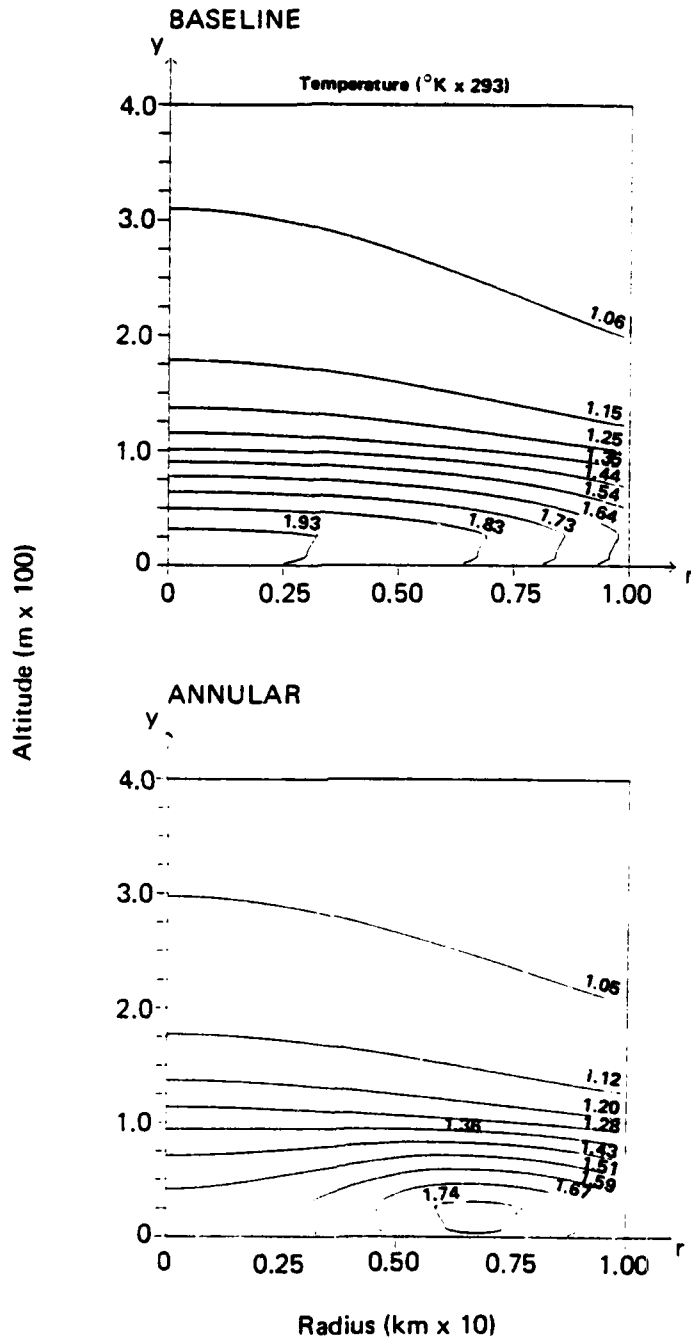


Fig. 12--Temperature contours of baseline and annular fires

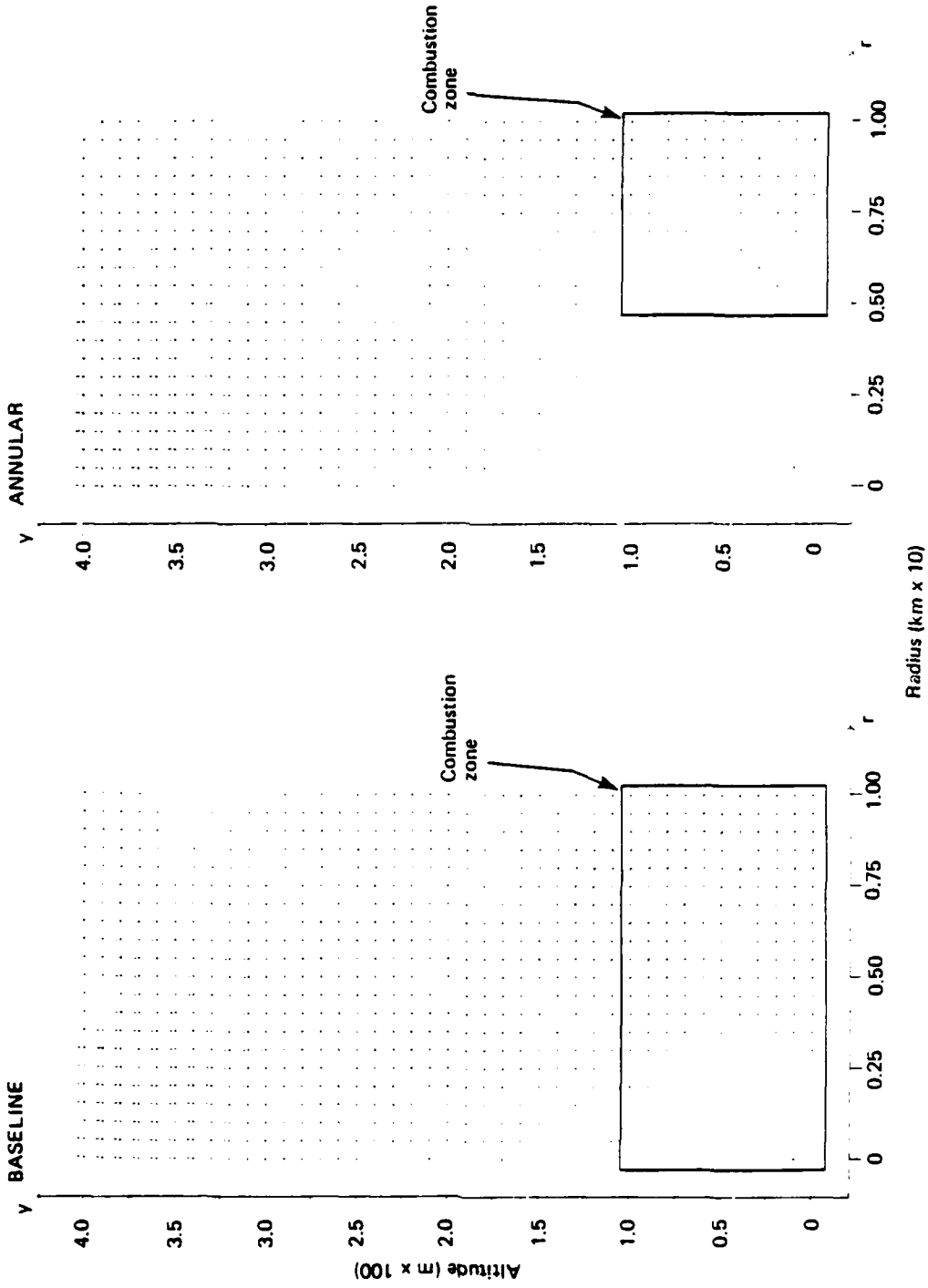


Fig. 13--Flow fields of baseline and annular fires

temperature field is thus qualitatively different from that of the baseline fire. Further, as Fig. 13 shows, winds in the annular fire do not blow toward the fire from both edges of the annulus. Rather, they blow inward (toward the symmetry axis) and upward at all points, as do those in the baseline fire. By means of heat convection and branding, annular fires might therefore be expected to spread inward.

Similarly, a cluster of separated large fires, such as could result from multiple nuclear bursts over a large city, might coalesce and engulf much of the intervening region. The patterns of airflow and fire spread envisioned for such situations are sketched in Figs. 14 and 15. The cluster of fires represented in Fig. 14 nearly forms an annulus, as indicated by the dashed circles. The fire-wind flow

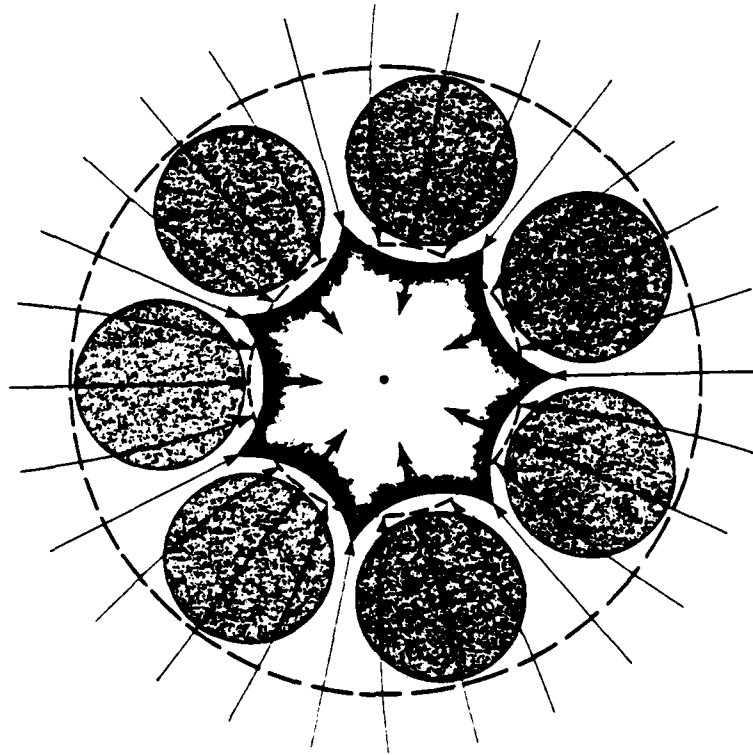


Fig. 14--Radial airflow and fire spread patterns suggested for annular cluster of large area fires

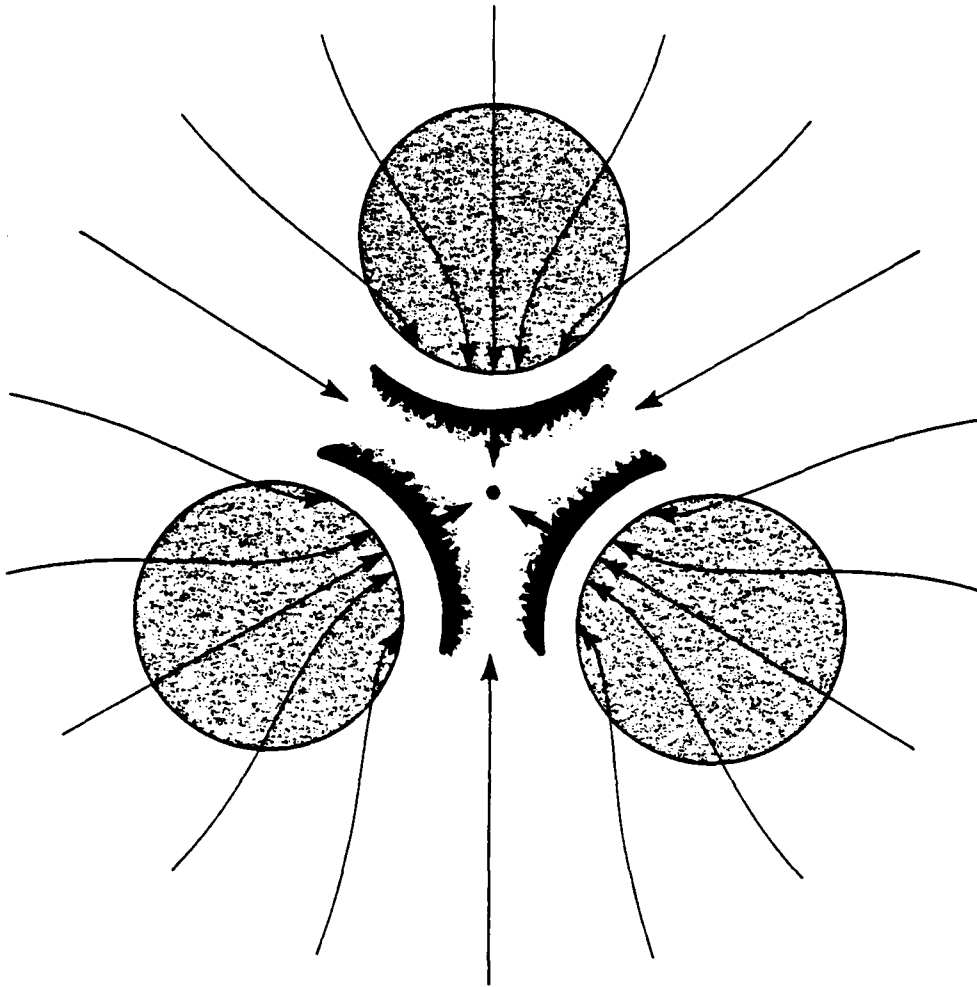


Fig. 15--Radial airflow and fire spread patterns suggested for cluster of three large area fires

should thus be nearly axisymmetric and of the type illustrated in Fig. 13. Correspondingly, the fire should spread inward, and may ultimately cover much of the area inside the inner dashed curve. As shown in Fig. 15, the flow and spread patterns for asymmetric clusters should be less axisymmetric, but still basically inward. Such hypotheses could presumably be verified by extensive, three-dimensional numerical computations.

We now consider a variation of heat release rate with altitude. Specifically, we compare the results for the baseline [$q(r, y)$ defined by Eq. (4)] with those for the more uniform case where q is independent of y --and hence constant. That comparison, summarized in Table 5, reflects the results illustrated in Figs. 8 and 11. For less concentrated heat releases, temperatures are lower, but pressure drops and velocities are greater.

Table 5
VELOCITY, TEMPERATURE, AND PERTURBATION PRESSURE MAXIMA
FOR BASELINE AND UNIFORM FIRES

Type of Fire	$q(r, y)$	u_{\max} (m/sec)	T_{\max} (°K)	P_{\max} (psi)	v_{\max} (m/sec)
Baseline	$\left. \begin{array}{ll} 1.6 & \text{for } 0 \leq y \leq 0.25 \\ 1.6 \left(\frac{4}{3}\right) (1 - y) & \text{for } 0.25 \leq y \leq 1 \\ 0 & \text{for } y \geq 1 \end{array} \right\}$	37.2	570	0.156	3.26
Uniform	1.0	50.9	497	0.231	7.19

DEPENDENCE ON TURBULENCE AND RADIATION

Finally, we consider the variation of the large-fire environment with the eddy coefficients M_1 , K_1 and the graybody radiation parameter k^* . The results are summarized in Figs. 16 through 23.

The dependence of fire winds and temperatures on eddy coefficients is demonstrated by Figs. 16, 17, and 18. As expected, the maximum radial velocity decreases with an increase in the eddy coefficient for momentum transfer (Fig. 16). Correspondingly, the maximum temperature increases. An increase in the eddy coefficient for heat transfer might also decrease temperatures, and hence increase velocities. In Fig. 17, such variations are shown to occur, but so weakly that both T_{\max} and u_{\max} must be considered almost independent of K_1 . When M_1 and K_1 are varied together, as in Fig. 18, the results are thus similar to those with only M_1 varied (cf. Fig. 16).

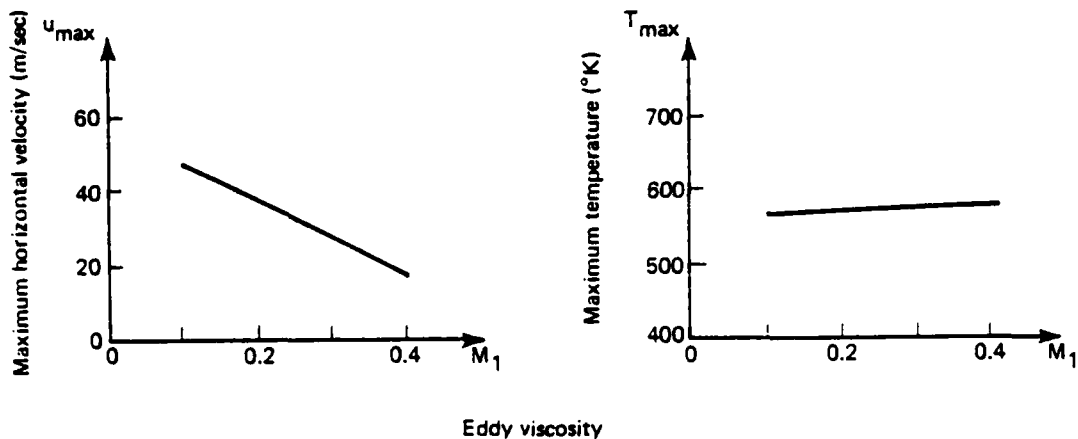


Fig. 16--Dependence of maximum radial velocity and temperature on eddy coefficient for momentum transfer

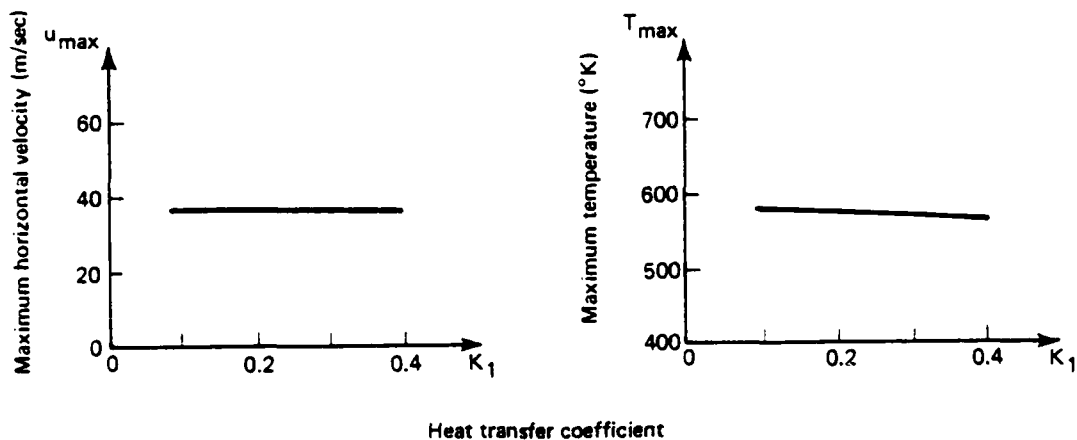


Fig. 17--Dependence of maximum radial velocity and temperature on eddy coefficient for heat transfer

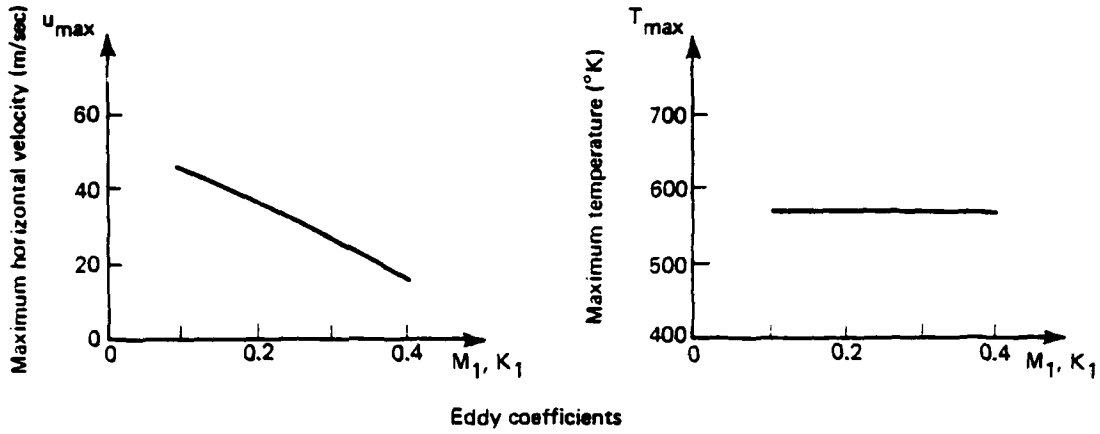


Fig. 18--Dependence of maximum radial velocity and temperature on eddy coefficients

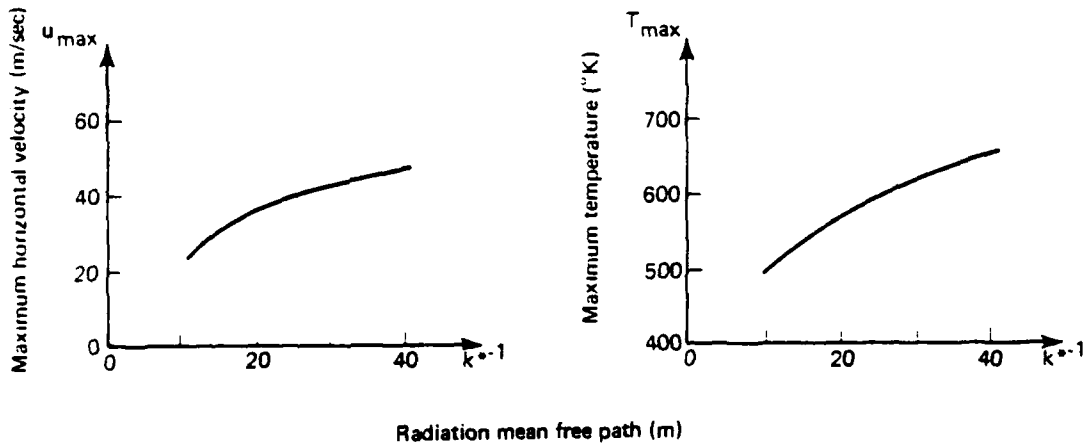


Fig. 19--Dependence of maximum radial velocity and temperature on radiation mean free path

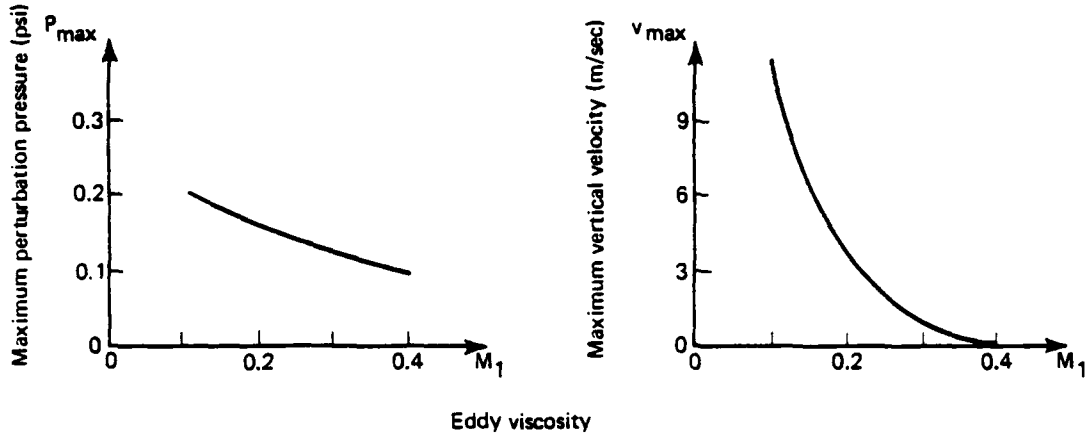


Fig. 20--Dependence of maximum perturbation pressure and vertical velocity on eddy coefficient for momentum transfer

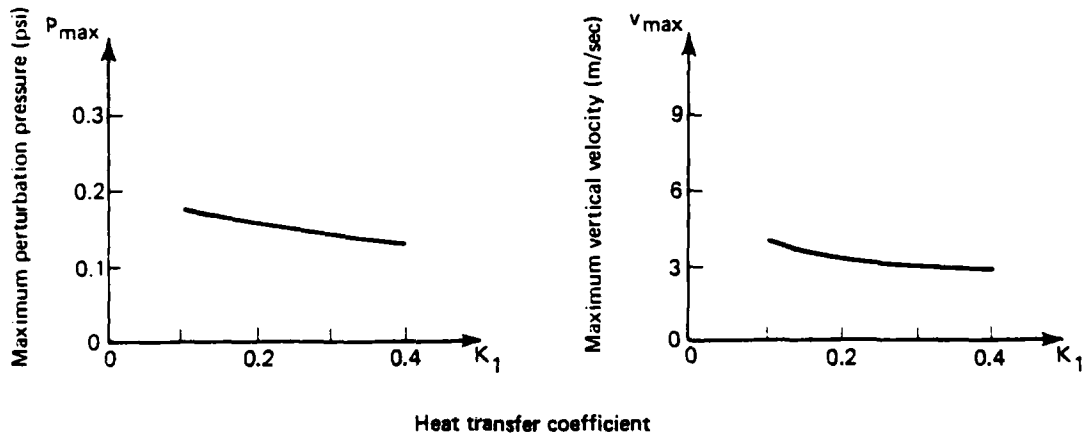


Fig. 21--Dependence of maximum perturbation pressure and vertical velocity on eddy coefficient for heat transfer

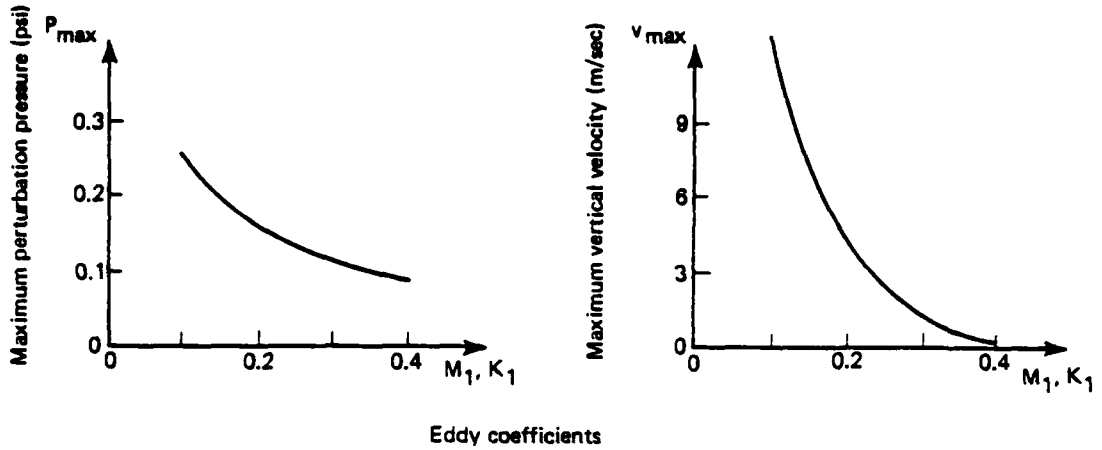


Fig. 22--Dependence of maximum perturbation pressure and vertical velocity on eddy coefficients

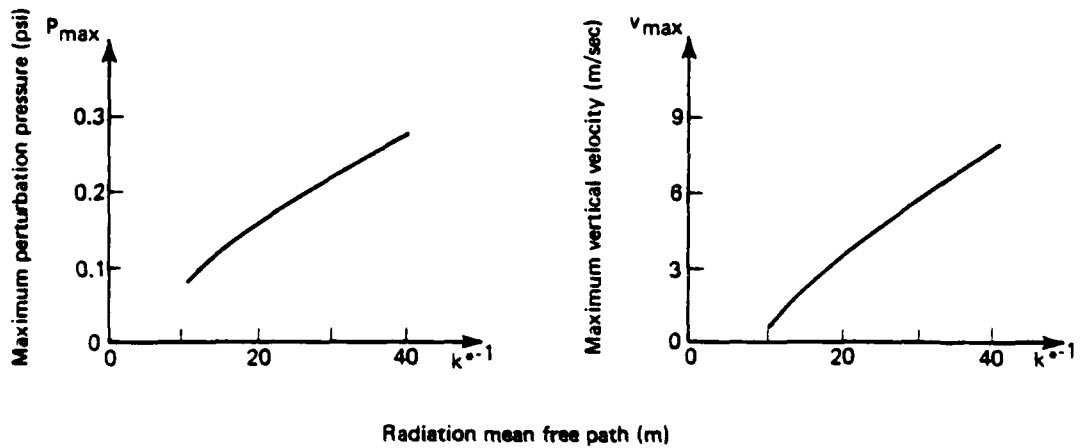


Fig. 23--Dependence of maximum perturbation pressure and vertical velocity on radiation mean free path

Figure 19 indicates the dependence of fire winds and temperatures on the radiation parameter k^* . Both winds and temperatures increase with k^{*-1} --as expected, since the effective scale for radiative energy loss σ is proportional to k^* [see Eqs. (1) and (2)].

The dependence of pressure drops and vertical velocities on M_1 , K_1 , and k^* is shown by Figs. 20 through 23. As in previous comparisons (Figs. 6 through 11 and 16 through 19), variations in those parameters are strongly correlated with variations in radial velocity.

III. MODEL CITY ANALYSIS

Using the turning-region analysis of Part I, we obtain predictions of the large-fire environment for three model U.S. cities. City W is lightly built-up, and intended to represent new, sprawling cities. City E is heavily built-up, and intended to represent old, congested cities. City M is of intermediate building density. For each city, two cases are considered: a baseline fire and one modified by blast. In all cases, however, the fire radius is taken to be 12 km--believed to be about the size of the fire that would result from a 1 Mt near-surface nuclear burst [Johnson and Larson, 1982].

DEFINITION

Few metropolitan areas are axisymmetric. Nevertheless, most cities have a main business district with high-rise office and apartment buildings, surrounded by lower density tracts. Our model cities reflect that situation. Each has three regions: a tall central city; a residential/industrial belt of intermediate height around the central city; and a low, primarily residential outer belt. We refer to those regions as the central city, inner belt, and outer belt, respectively.

Table 6 summarizes the geometry of the model cities, including both the horizontal extent of each city region, the corresponding vertical extent N_s (average number of stories), and the building density f_{Bu} . We assume that the top and bottom stories are each 4 m high and all other stories are 3 m high.[†] The values for building density f_{Bu} in each model city region are comparable to those in the DCPA manual, and thus considered representative of U.S. cities [DCPA, 1973]. The values are all less than 0.45, which approximates the densely packed Hamburg districts devastated in the firestorm of 1943 (see Part I).

[†]The average heights of the outer-belt buildings in cities W and E are taken to be 6 and 10 m, respectively.

Table 6

SIZE AND DENSITY OF MODEL CITY REGIONS

Region	City		
	W	M	E
<i>Radial Dimension (km)</i>			
Central city	0-1	0-1	0-2
Inner belt	1-3	1-4	2-6
Outer belt ^a	3-12	4-12	6-12
<i>Average Number of Stories, N_s</i>			
Central city	8	12	16
Inner belt	3	4	6
Outer belt	1.5	2	2.5
<i>Building Density, f_{Bu}^b</i>			
Central city	0.30	0.35	0.40
Inner belt	0.25	0.30	0.35
Outer belt	0.15	0.20	0.25

^aRadius of outer belt may extend beyond 12 km, but the fire does not.

^bCalculated as the ratio of land area covered by buildings to total land area.

The corresponding geometry of the baseline model-city fire simulations is summarized in Figs. 24 through 26. In each figure, the shaded area represents the fuel zone (defined by Table 6) and the hatched area represents the remainder of the combustion zone. The total height of the combustion zone in the outer belt is assumed to be five times the fuel zone height.[†] Since turbulence will break the flame envelope and tend to keep flame heights uniform [Thomas, 1963], the ratio of combustion zone height to fuel zone height in the central city is taken as only half as much; the combustion zone height in the inner belt is assumed to be the average of the other two.

[†]As noted in Part I, a combustion-zone-height to fuel-zone-height ratio of about five was observed for the largest Flambeau fires and the 1943 Hamburg firestorm.

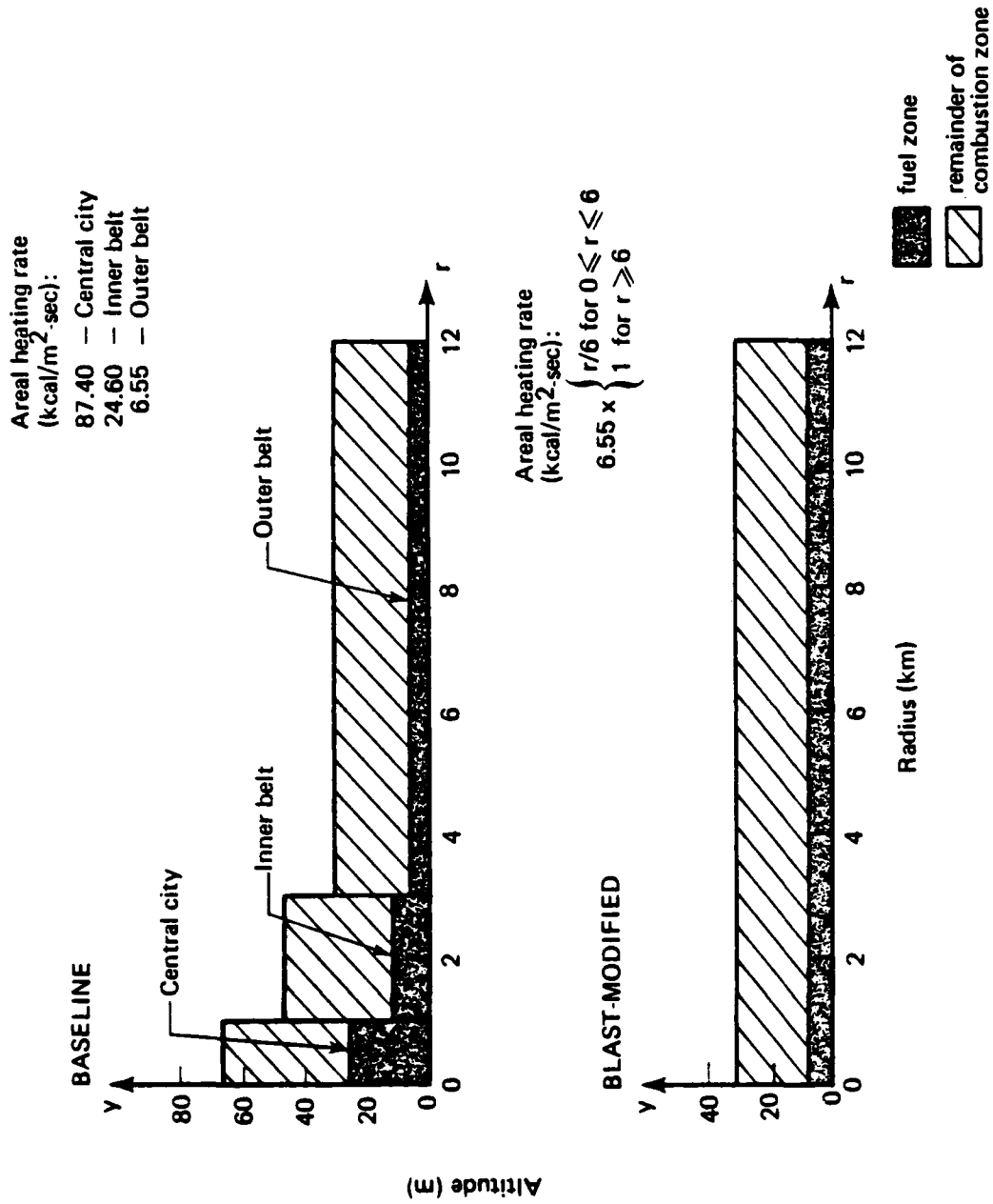


Fig. 24--Fire schematic for baseline and blast-modified city W

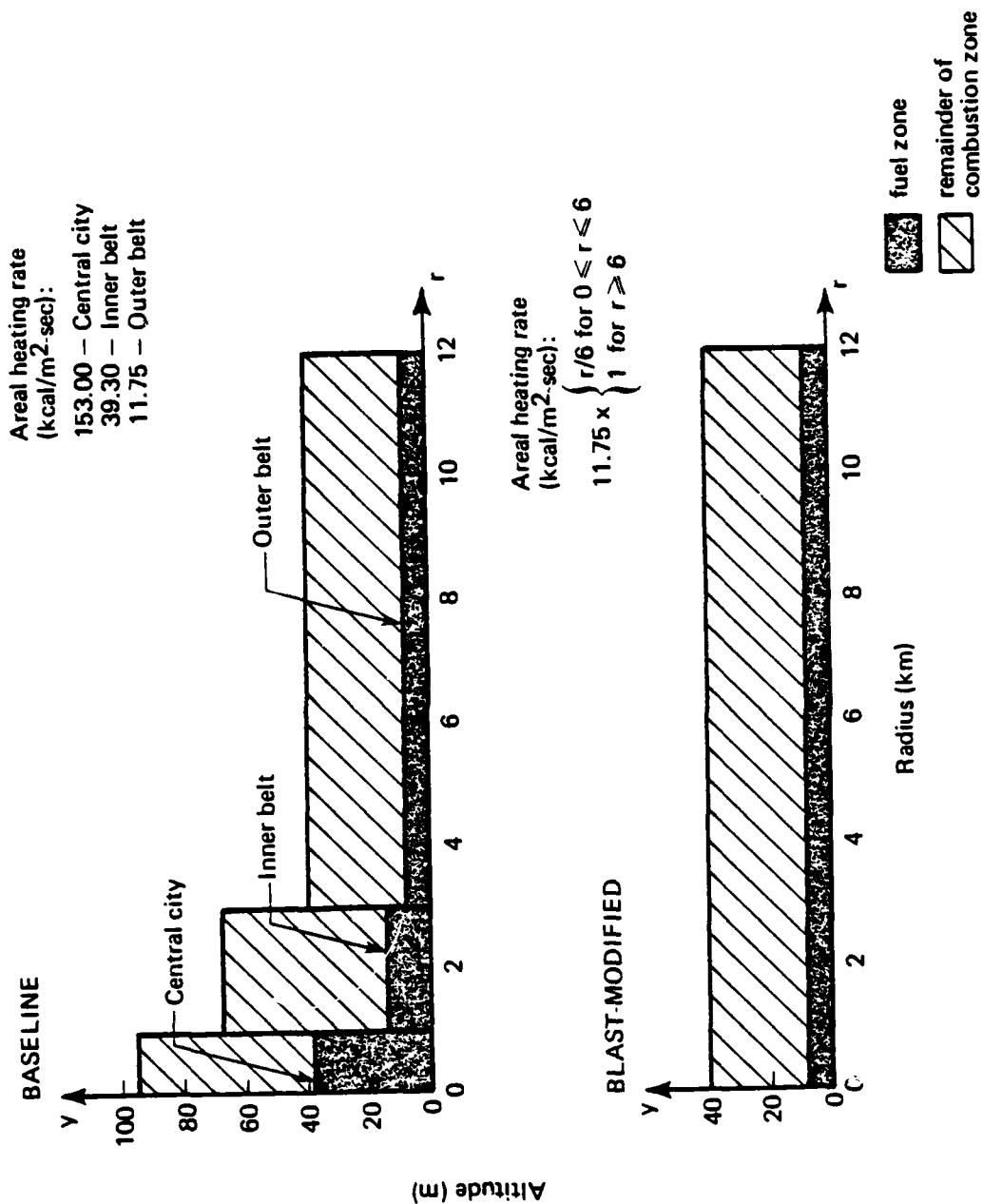


Fig. 25--Fire schematic for baseline and blast-modified city M

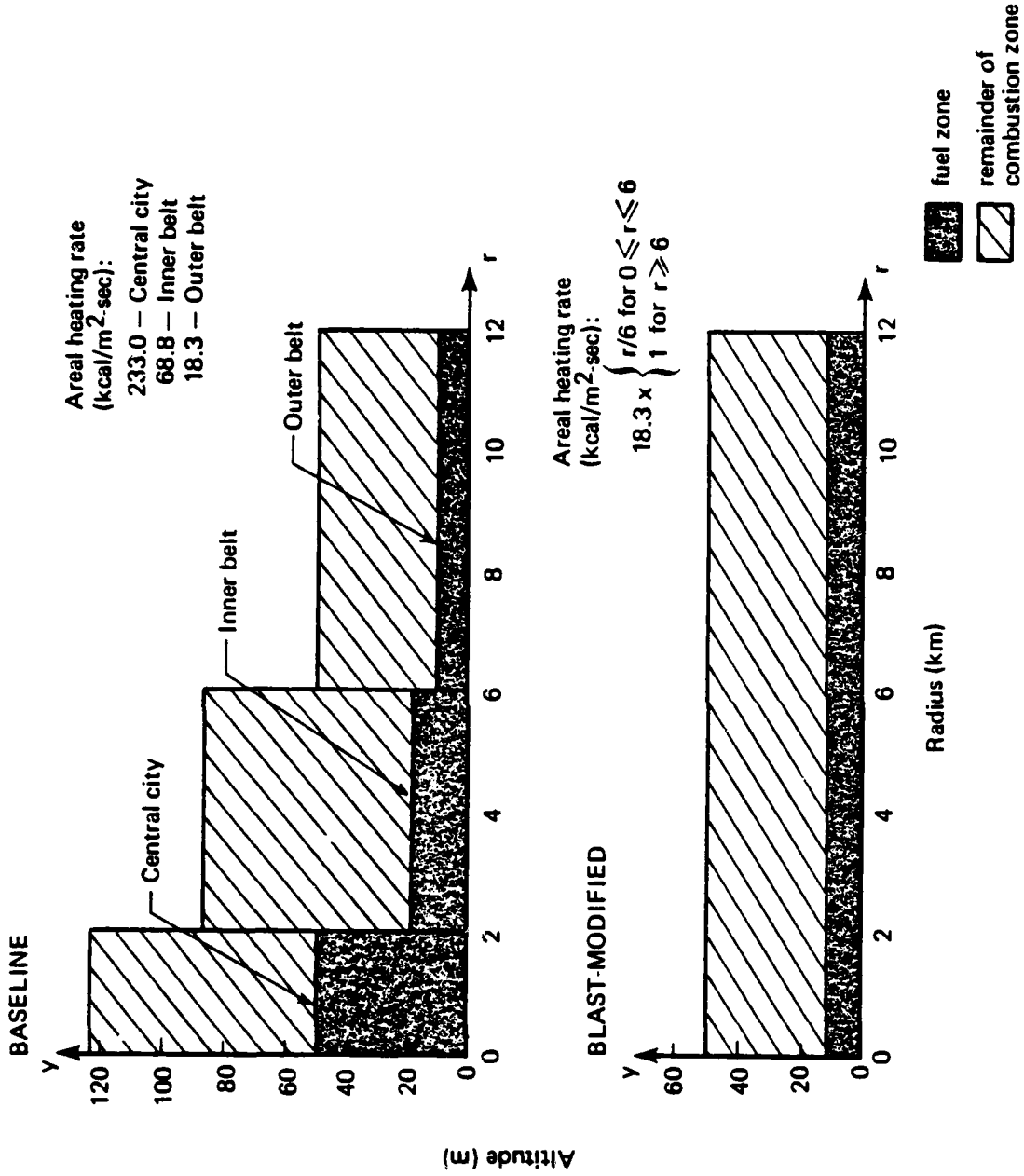


Fig. 26--Fire schematic for baseline and blast-modified city E

The baseline areal heating rates (see Figs. 24 through 26) for the central city, inner belt, and outer belt are estimated as follows. The average fuel loading per story L_s in each city region is defined in Table 7. The corresponding average total areal fuel loading L_T is then computed from

$$L_T = L_s \times N_s \times f_{Bu} , \quad (7)$$

with values for N_s and f_{Bu} taken from Table 6. The resulting total-loading estimates are also listed in Table 7. The central cities and outer belts are somewhat taller and shorter, respectively, than were those in the devastated Hamburg districts, and have higher and lower total loadings. The inner belts are defined to be on the order of the

Table 7
FUEL LOADING IN MODEL CITY REGIONS

Region	City		
	W	M	E
<i>Average Fuel Loading per Story, L_s^a</i> <i>(lb/ft²-story)</i>			
Central city	20	20	20
Inner belt	18	18	18
Outer belt	16	16	16
<i>Total Areal Fuel Loading, L_T</i> <i>(lb/ft²)</i>			
Central city	48.0	84.0	128.0
Inner belt	13.5	21.6	37.8
Outer belt	3.6	6.4	10.0

^aBased on DCPA [1973] estimates, which take into account both the structures and their contents.

Hamburg districts, but less built-up. Similarly, the inner-belt loadings tend to be somewhat less than the 32 lb/ft² estimated for Hamburg [DCPA, 1973].

Finally, in each model city region, the fire is assumed to consume 90 percent of the available fuel in 3 hr. The areal heating rate Q_A is thus computed from

$$Q_A = L_T \times \left(\frac{90\%}{3 \text{ hr}} \right) \times E_B , \quad (8)$$

with L_T taken from Table 7 and E_B , the average heat released per unit weight of combustibles, taken as [DCPA, 1973]

$$E_B = 8000 \text{ Btu/lb} . \quad (9)$$

For the blast-modified fire simulations represented in Figs. 24 through 26, the weapon burst is assumed to occur over the city center, leveling many buildings in the central city and inner belt. A centered, low-level burst of a 1 Mt weapon would produce such an effect, with blast damage extending out to radii on the order of 6 km (and fire to around 12 km) [Johnson and Larson, 1982]. The height of the fuel zone is thus simply assumed constant and equal to its outer-belt value in the baseline case. The total height of the combustion zone is chosen similarly.

The blast-modified areal heating rates are not considered independent of radius, however. In each case, the combustibles of the central city and inner belt would be spread radially to some degree by the blast, and piled up in a debris field. Since some combustibles in that zone may be buried under layers of nonflammable materials (e.g., concrete, brick, metal), the areal heating rate is not expected to be correspondingly higher, and may in fact be relatively small. We consider such a situation in each simulation. It is assumed that the areal heating rate is zero at the fire center, increases linearly with radius over the debris zone, and equals its baseline outer-belt value for radii greater than 6 km.

In each model city simulation, the required data set consists of the geometric parameters R and H, the heating rate parameters QH and $q(r, y)$, the radiation coefficient σ , and the turbulence coefficients M_1 and K_1 . Once R, H, and QH are chosen, the parameter A in Eqs. (1) is determined along with the radial velocity scale U from Eqs. (2).

In all cases, the fire radius R is taken to be 12 km. However, since the combustion zone height is not constant in the baseline cases, there are no natural choices for H. The values for H listed in Table 8

Table 8
PARAMETERS IN MODEL CITY SIMULATIONS

Parameter	City		
	W	M	E
<i>Fundamental Parameters</i>			
R (km)	12	12	12
H (m)	40	60	100
QH (kcal/m ² -sec)	9.50	14.25	23.75
<i>Auxiliary Parameters</i>			
U (m/sec)	33.6	33.6	33.6
A	0.347	0.521	0.868
σ	0.264	0.264	0.264
M_1, K_1	0.125	0.125	0.125

are arbitrarily selected as characteristic of the baseline geometry in Figs. 24 through 26. The values of the heating rate scale QH in Table 8 are chosen likewise.

Table 8 also lists the corresponding values of U and A, calculated using Eqs. (2).[†] From Eqs. (2) and (3), σ depends on H, QH, and the radiation mean free path k^{*-1} ; and M_1 depends on R, U, and the dimensional eddy viscosity \mathcal{E}_1/ρ_a . As in our earlier simulation

[†]The value of the dimensionless perturbation pressure scale δ is 1.44×10^{-2} for all cases.

(Part I), we take $k^{*-1} = 20$ m, $\xi_1/\rho_a = 5.04 \times 10^4$ m²/sec, and $M_1 = K_1$. The values we use for σ , M_1 , and K_1 [from Eqs. (2) and (3)] are also listed in Table 8.

Finally, we use the heating rate distributions $q(r, y)$ defined in Figs. 27 through 29. The heating rate is taken to be constant in the fuel zone of each baseline city region, and to decrease linearly with altitude over the remainder of the combustion zone, approaching zero at the top of that zone. Subject to the choices of QH in Table 8, the fuel zone values of $q(r, y)$ are selected so that the resulting areal heating rates

$$\int_0^{\infty} QH q(r, y) dy$$

are equivalent to those in Figs. 24 through 26. The blast-modified forms of $q(r, y)$ are developed likewise.

SIMULATION RESULTS

The results of the model city simulations are summarized by Table 9, which compares predicted baseline and blast-modified velocity, temperature, and pressure maxima. As expected, the predictions for the baseline simulations are all larger than those for the blast-modified simulations. The differences are great for the temperature, pressure, and vertical velocity maxima, but small for the radial velocity maxima (10 percent or less). Therefore, the winds and wind damage resulting from nuclear-weapon-ignited fires may be relatively insensitive to the degree of blast disruption of the fuel bed.

In any case, the predictions in Table 9 indicate that the winds generated by a large urban fire will in themselves constitute a major threat. Although most of the velocities in the table are less than hurricane force (more than 30 m/sec), it should be noted that those values represent *means*. Near street level, where fire winds will be channeled between buildings, wind speeds of hurricane force should

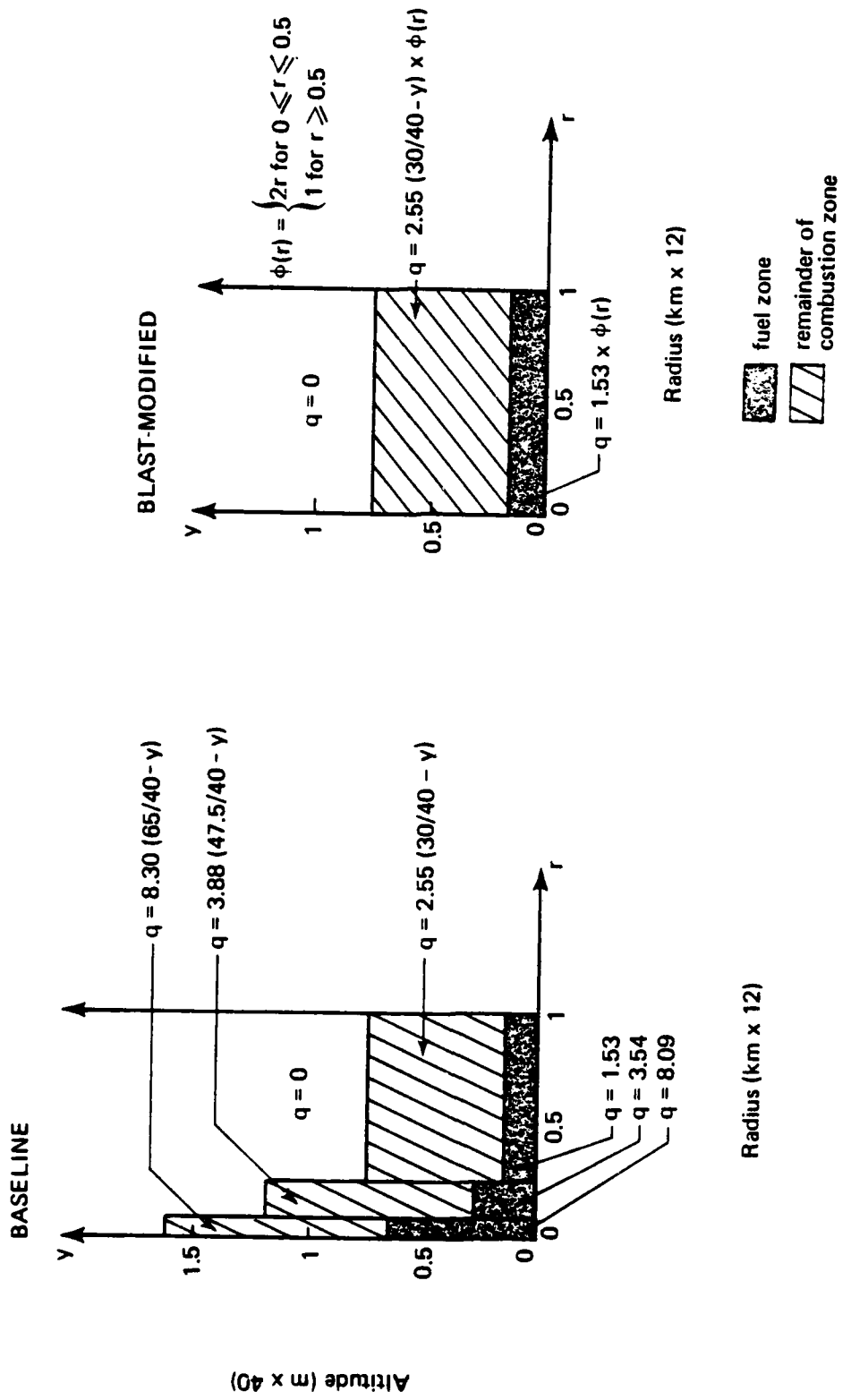


Fig. 27--Baseline and blast-modified heating rate spatial distributions for city W

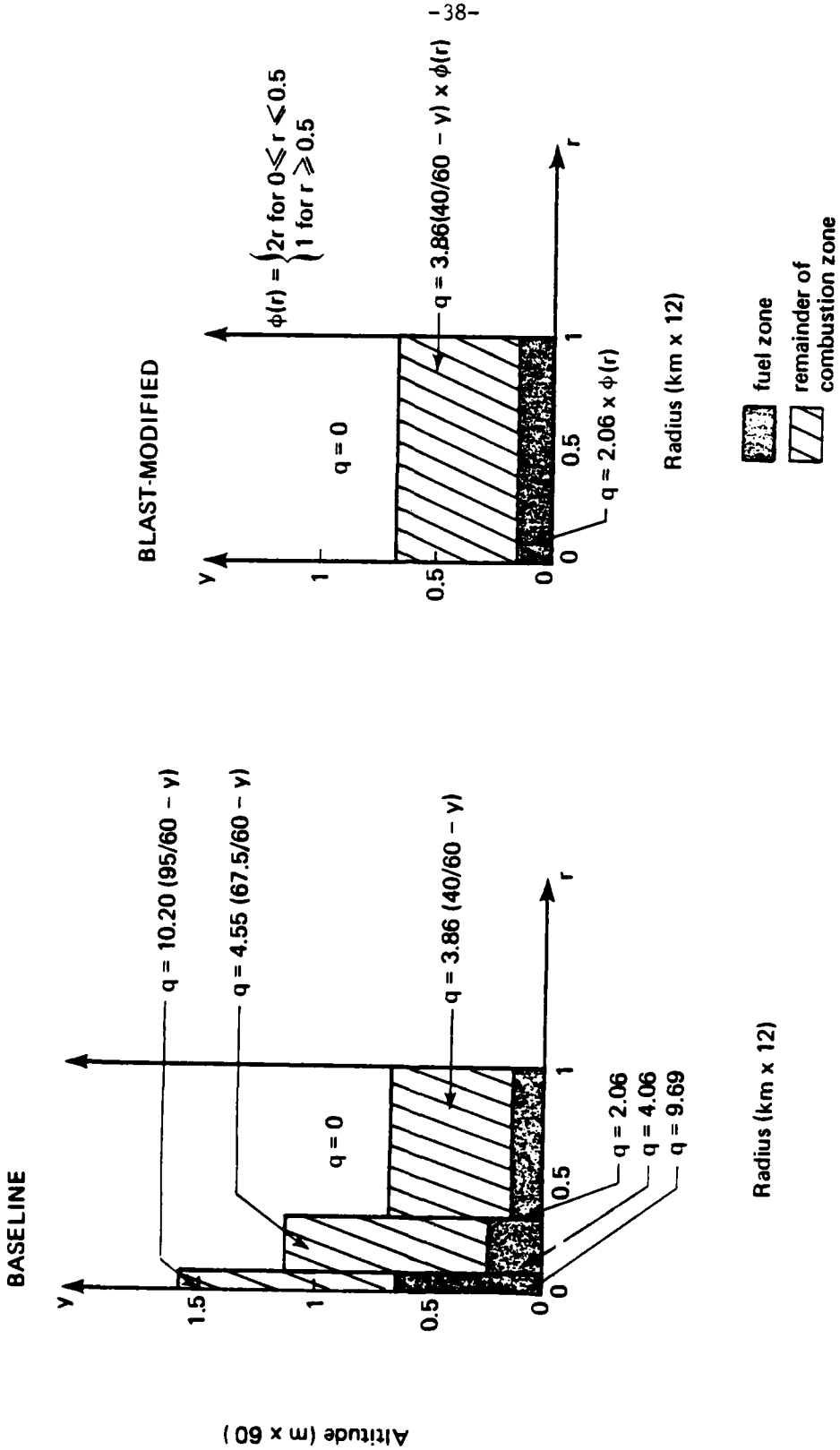


Fig. 28--Baseline and blast-modified heating rate spatial distributions for city M

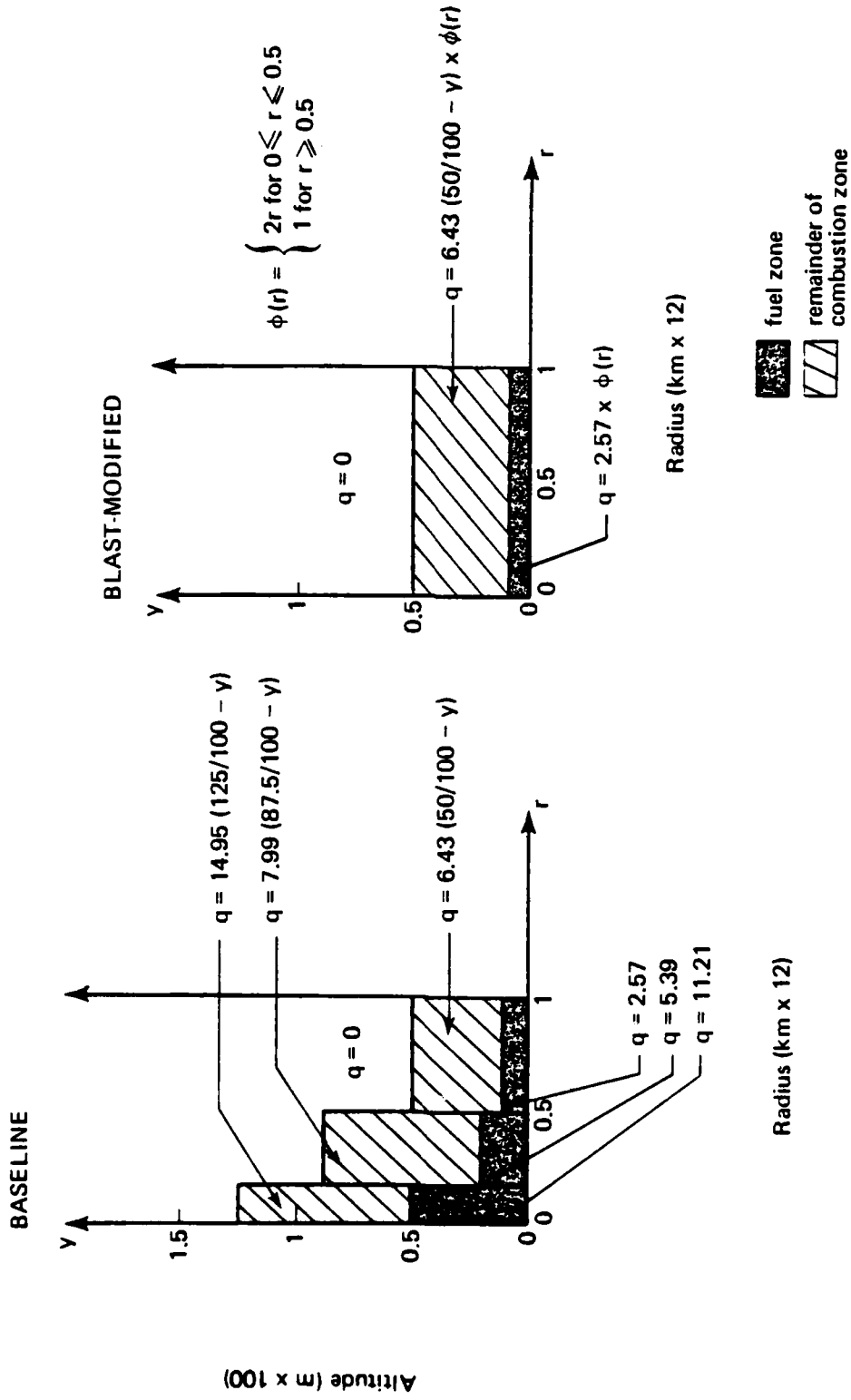


Fig. 29--Baseline and blast-modified heating rate spatial distributions for city E

Table 9

VELOCITY, TEMPERATURE, AND PERTURBATION PRESSURE
MAXIMA IN MODEL CITY SIMULATIONS

Simulation	City		
	W	M	E
<i>Maximum Radial Velocity, u_{max} (m/sec)</i>			
Baseline	20.2	26.3	39.0
Blast-Modified	17.9	23.9	28.5
<i>Maximum Temperature, T_{max} ($^{\circ}K$)</i>			
Baseline	577	619	704
Blast-Modified	455	485	510
<i>Maximum Perturbation Pressure, P_{max} (psi)</i>			
Baseline	0.056	0.113	0.271
Blast-Modified	0.011	0.044	0.076
<i>Maximum Vertical Velocity, v_{max} (m/sec)</i>			
Baseline	0.89	3.12	12.48
Blast-Modified	0.37	1.56	4.32

be considered typical. The winds may be even greater than those encountered in the Hamburg firestorm of 1943 (see Part I).

Also as expected, the velocity, temperature, and pressure predictions in Table 9 are all greatest for city E (the tallest and densest) and least for city W (the shortest and sparsest). For a given fire, therefore, the threat will be most severe for the most congested cities. In general, however, the shorter cities sprawl out over greater areas than do taller ones of comparable population, and are thus capable of supporting more widespread fires. Multiple weapon bursts can greatly increase the fire severity in such cities.

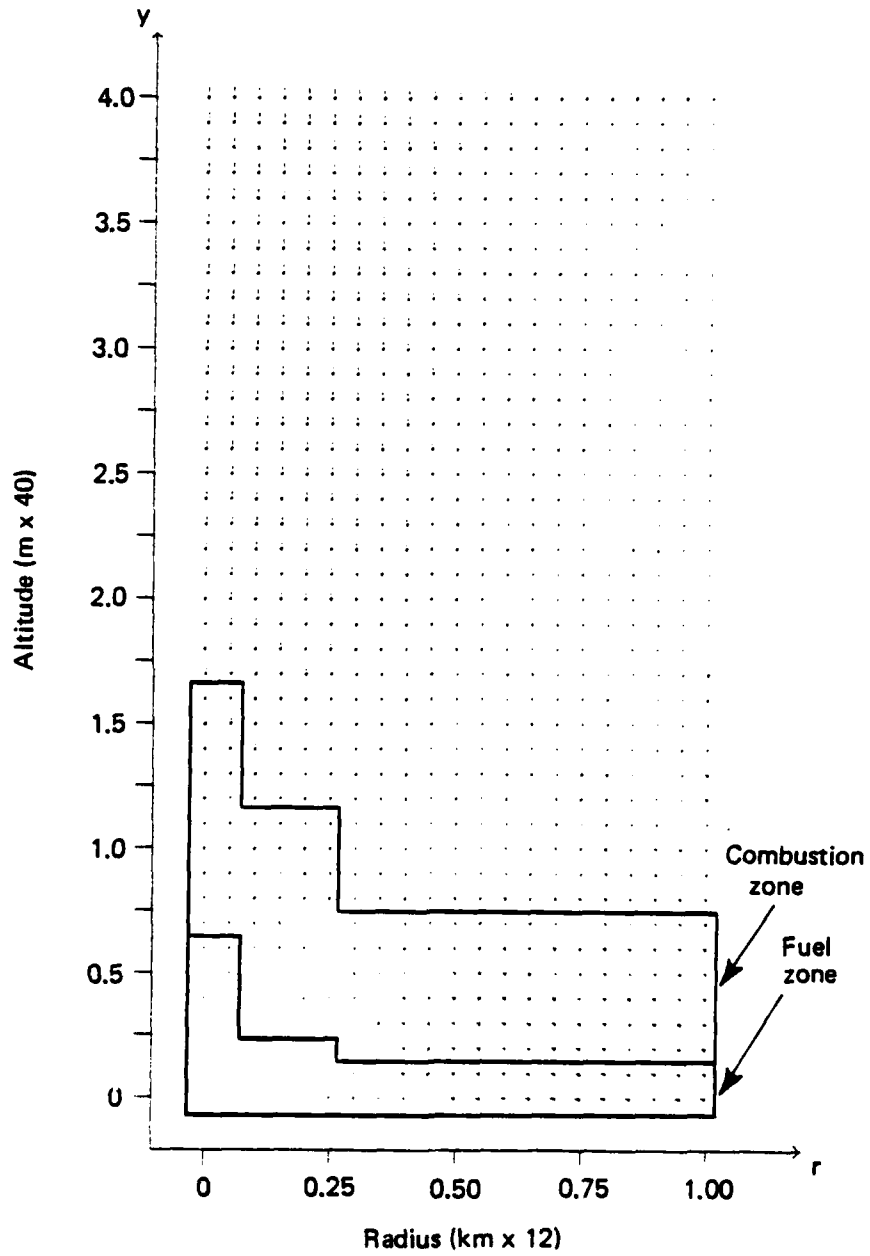
In the remainder of this subsection, we describe the results of the model city simulations in more detail. The vector plot

in Fig. 30 is representative of the flow fields obtained in the simulations. The flow in each case is everywhere inward (toward the symmetry axis) and upward, with the horizontal velocity typically maximum on the symmetry axis in the upper portion of the turning region. Figures 31 through 34 present, respectively, the temperature and pressure contours and radial and vertical velocity profiles for baseline and blast-modified simulations for city W. Similar sets of plots for cities M and E are presented in Figs. 35 through 38, and 39 through 42, respectively.

As expected (cf. Fig. 12), the baseline and blast-modified temperature contours in Fig. 31 are quite different. In the baseline case, the heat release is greatest at the center of the fire (see Fig. 27) and the temperature is maximum on the symmetry axis. At any fixed altitude, the variation of temperature with radius is roughly Gaussian. However, in the blast-modified case, the temperature maximum does not occur on the symmetry axis. Rather, since the heat release is greatly diminished and actually least in the fire center (Fig. 27), the temperature is maximum about halfway between the center and the periphery. Additionally, the temperature is uniformly reduced in the blast-modified case.

Figures 32 through 34 show little qualitative difference between the baseline and blast-modified pressures and velocities. The contour plots in Fig. 32 exhibit the same behavior, the pressure drop decreasing with an increase in either radius or altitude. Likewise, the profile plots in both Figs. 33 and 34 are qualitatively the same. The radial velocity is maximum at ground level on the periphery and the vertical velocity is maximum on the symmetry axis in the upper part of the turning region. At any specified altitude, the variation of vertical velocity with radius is again roughly Gaussian. Of course, as expected, certain quantitative differences are evident: pressure drops and velocities are all less in the blast-modified case than in the baseline case. However, the radial velocity differences are relatively small.

The comparisons in both Figs. 35 through 38 and Figs. 39 through 42 are similar to those in Figs. 31 through 34. The primary difference



Source: From calculation for baseline city W.

Fig. 30--Typical velocity field in model city simulations

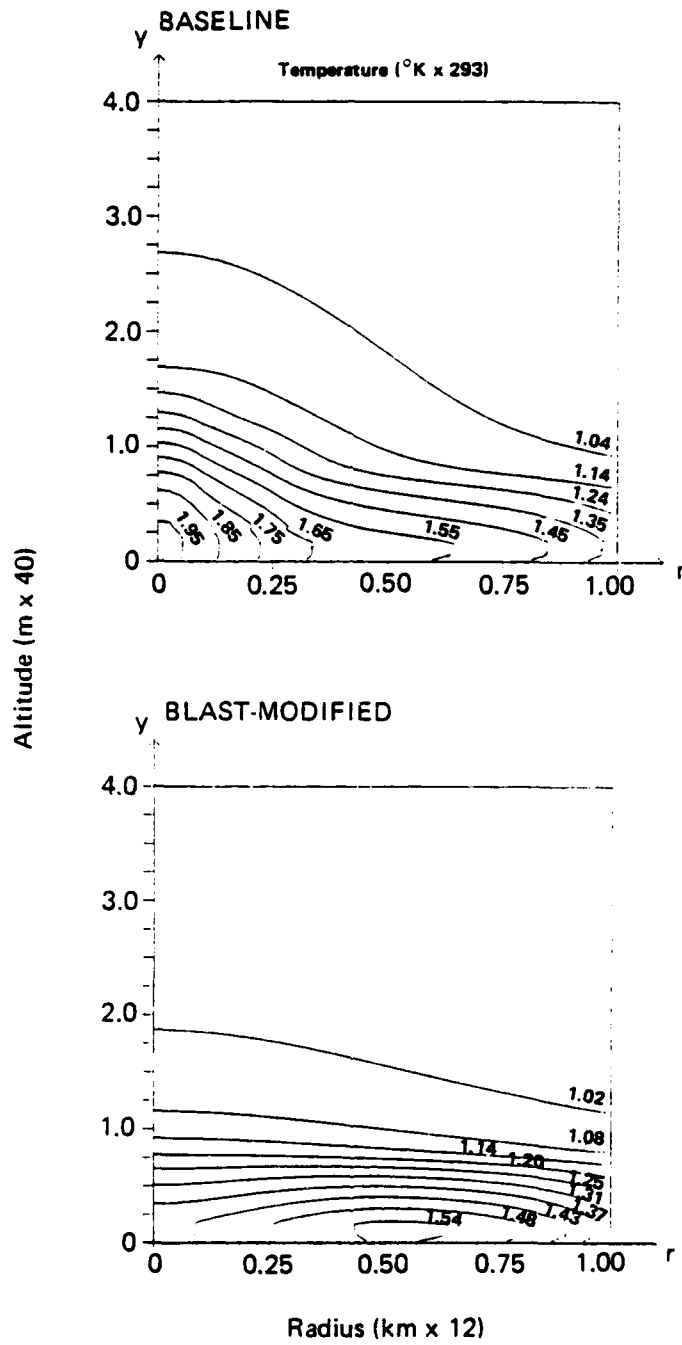
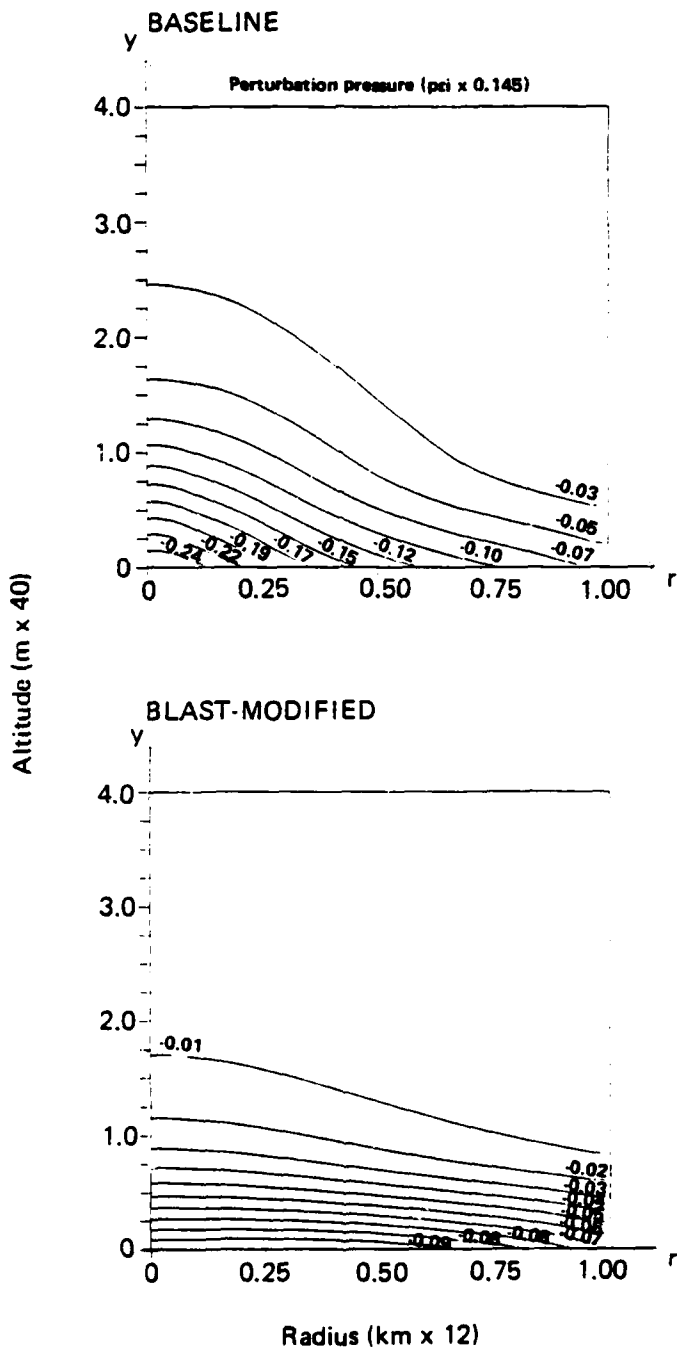


Fig. 31--Baseline and blast-modified temperature contours for city W



Note: Perturbation pressure is $\delta P_s (P + Ay)$; see footnote † on p. 6.

Fig. 32--Baseline and blast-modified pressure contours for city W

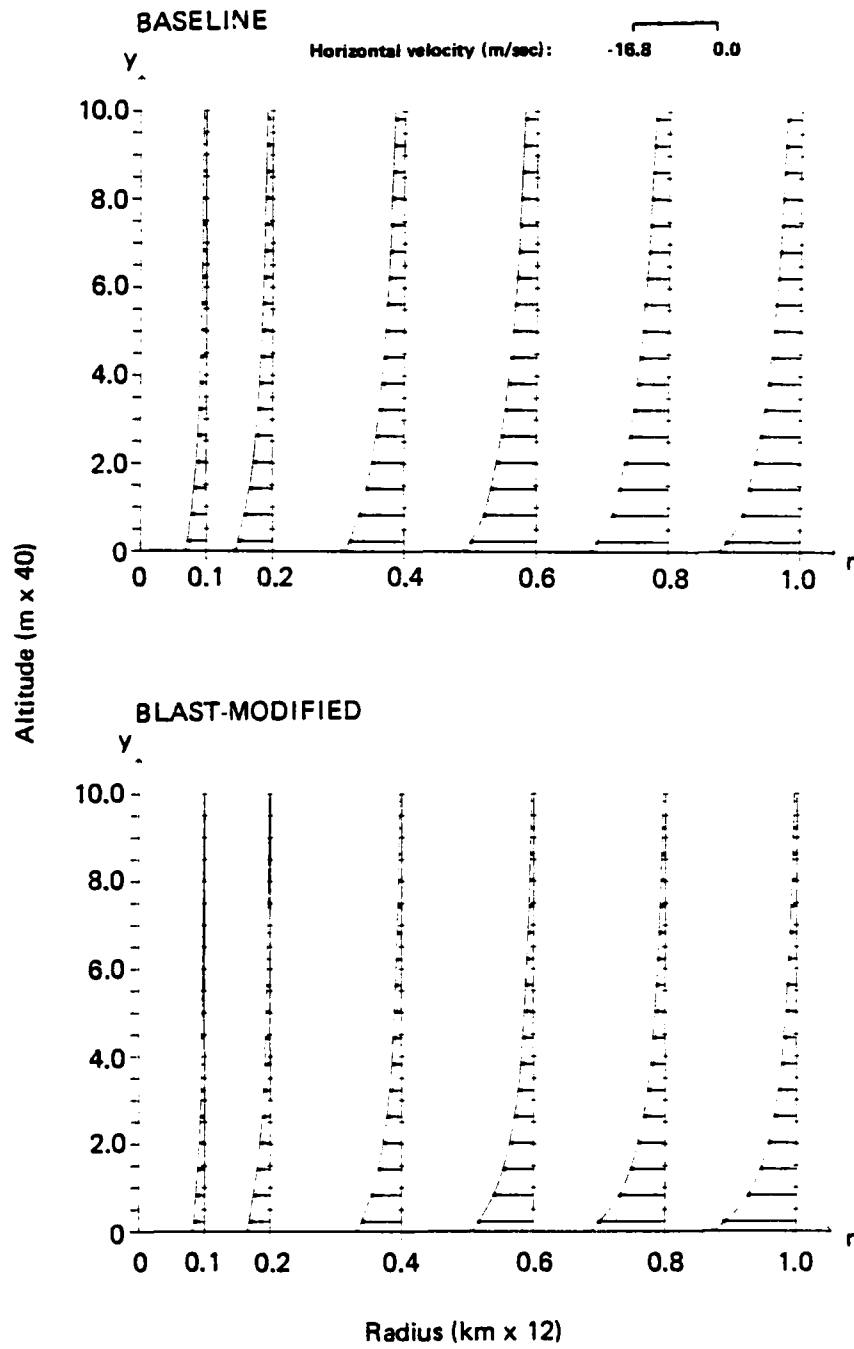


Fig. 33--Baseline and blast-modified radial velocity profiles for city W

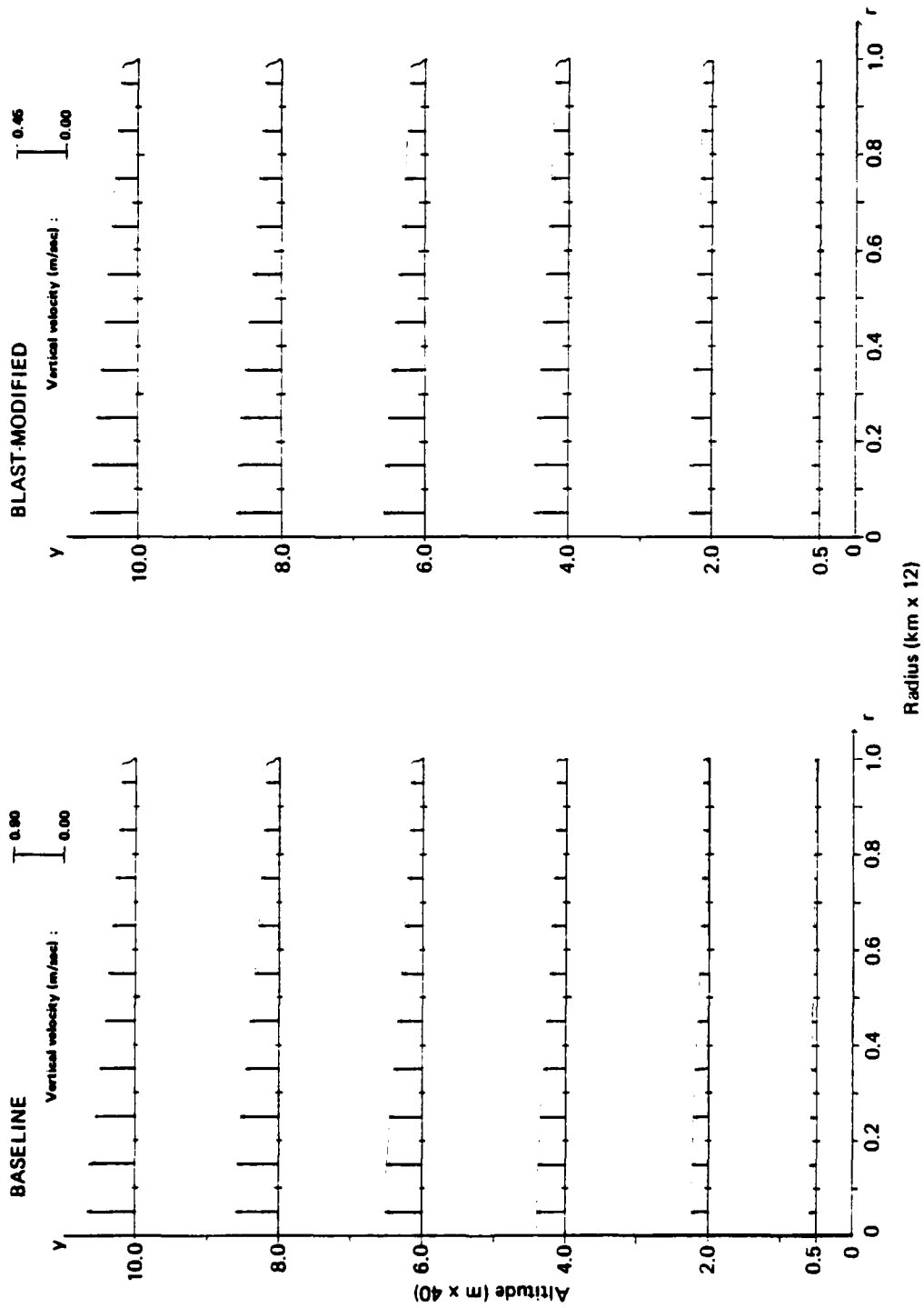


Fig. 34--Baseline and blast-modified vertical velocity profiles for city W

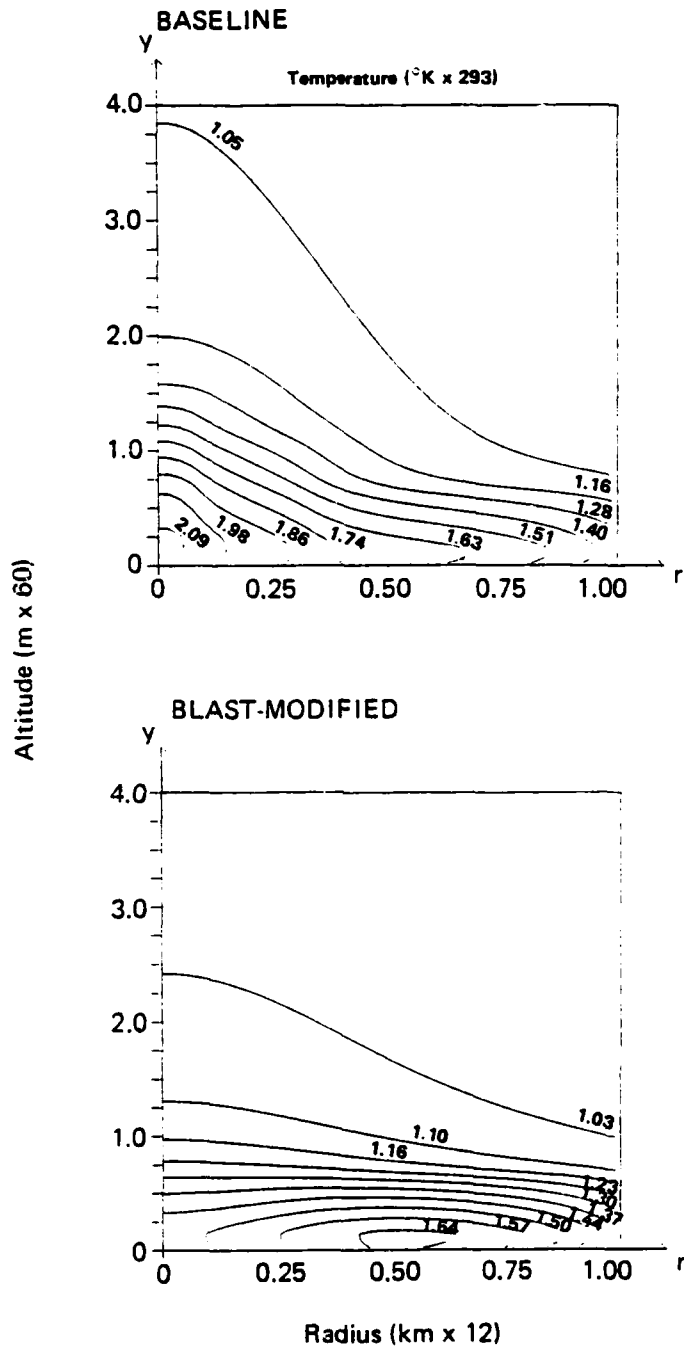


Fig. 35--Baseline and blast-modified temperature contours for city M

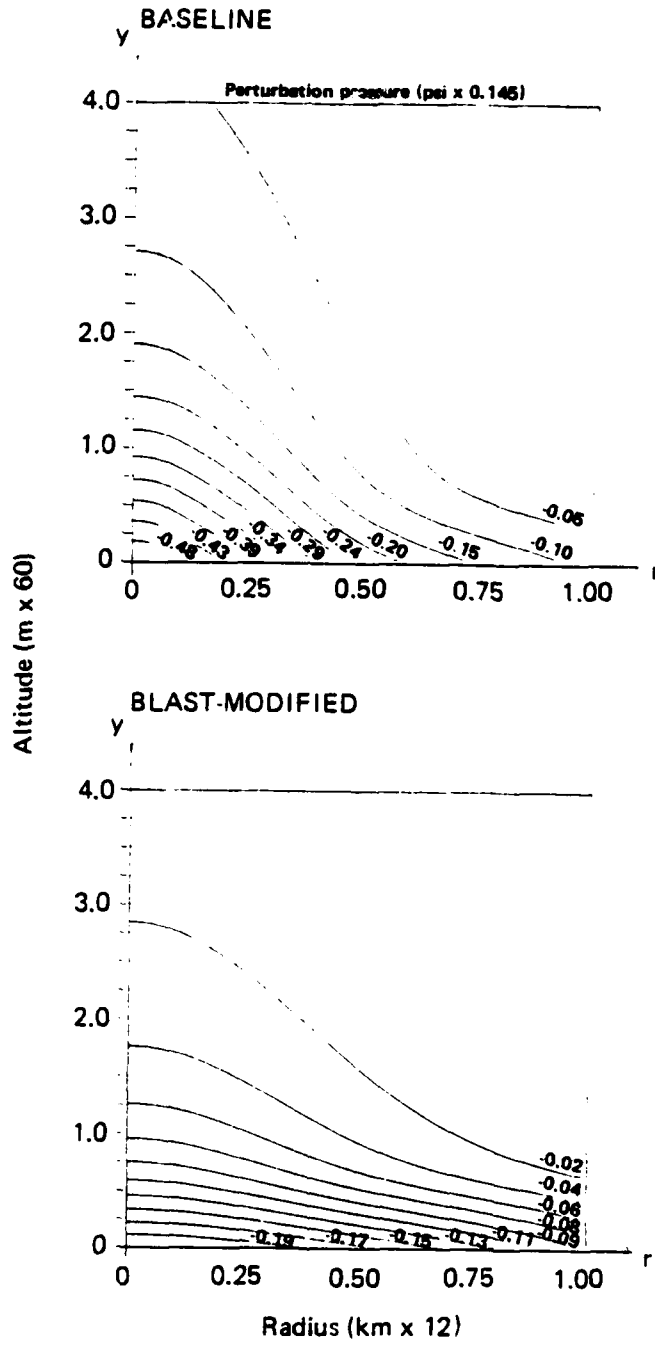


Fig. 36--Baseline and blast-modified pressure contours for city M

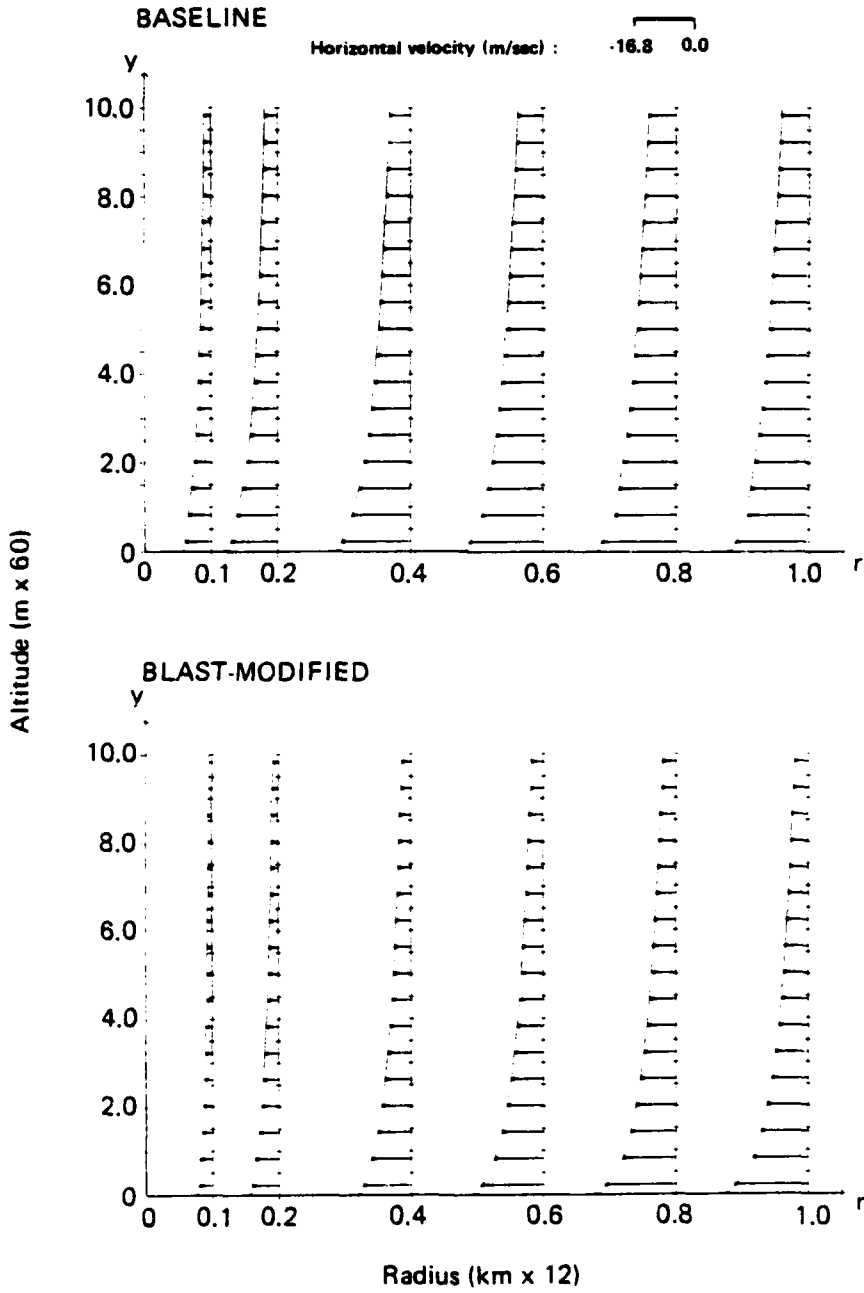


Fig. 37--Baseline and blast-modified radial velocity profiles for city M

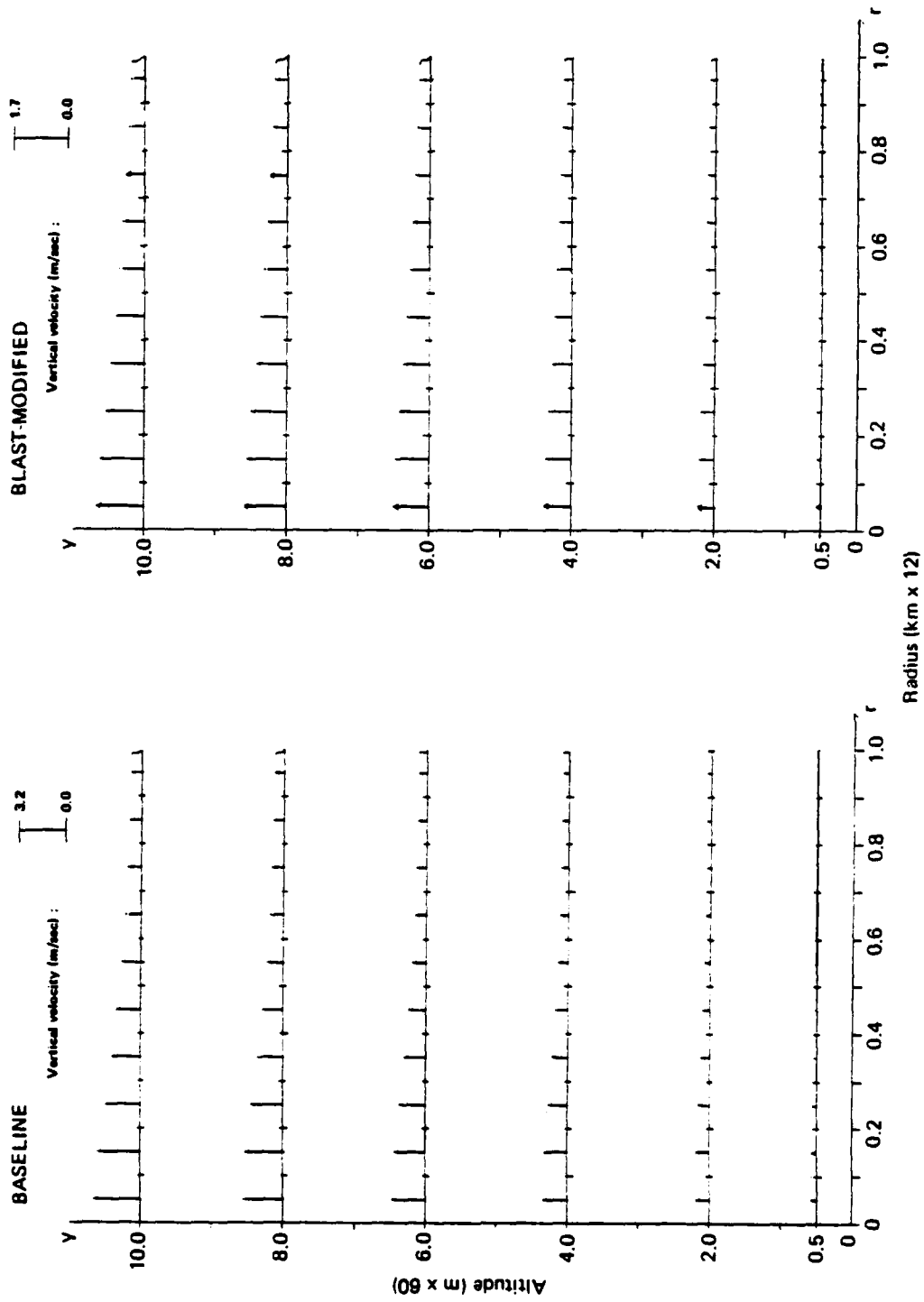


Fig. 38--Baseline and blast-modified vertical velocity profiles for city M

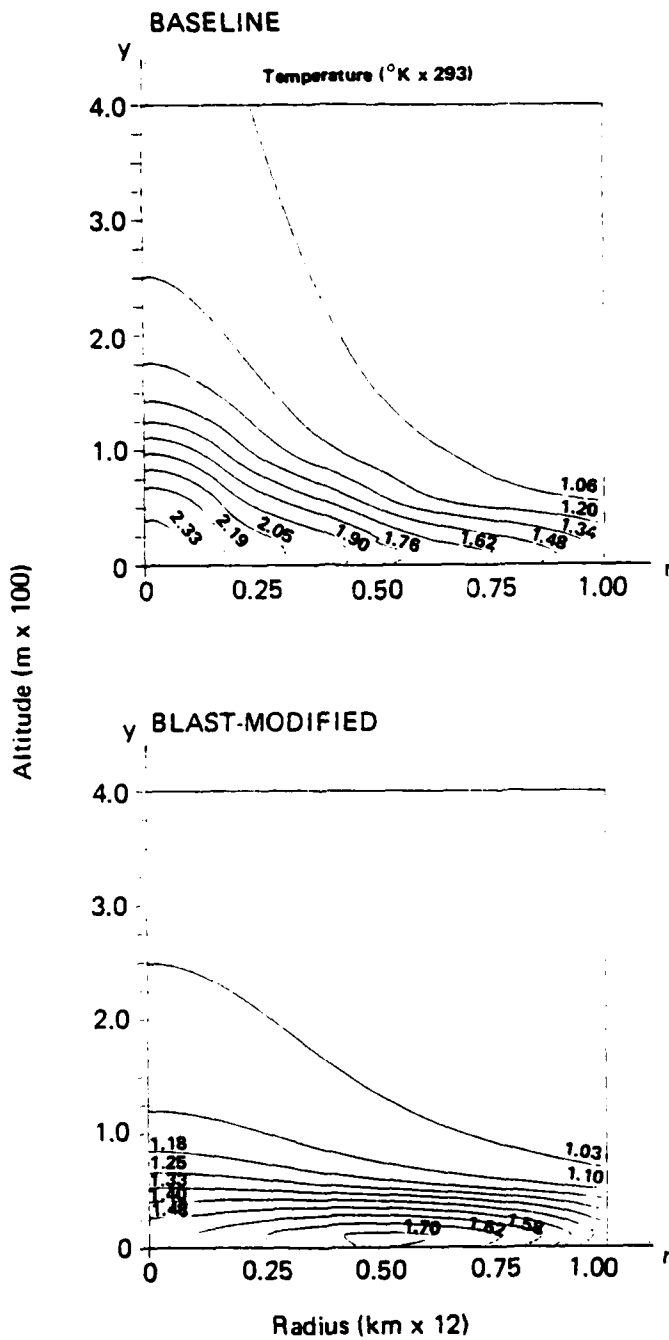


Fig. 39--Baseline and blast-modified temperature contours for city E

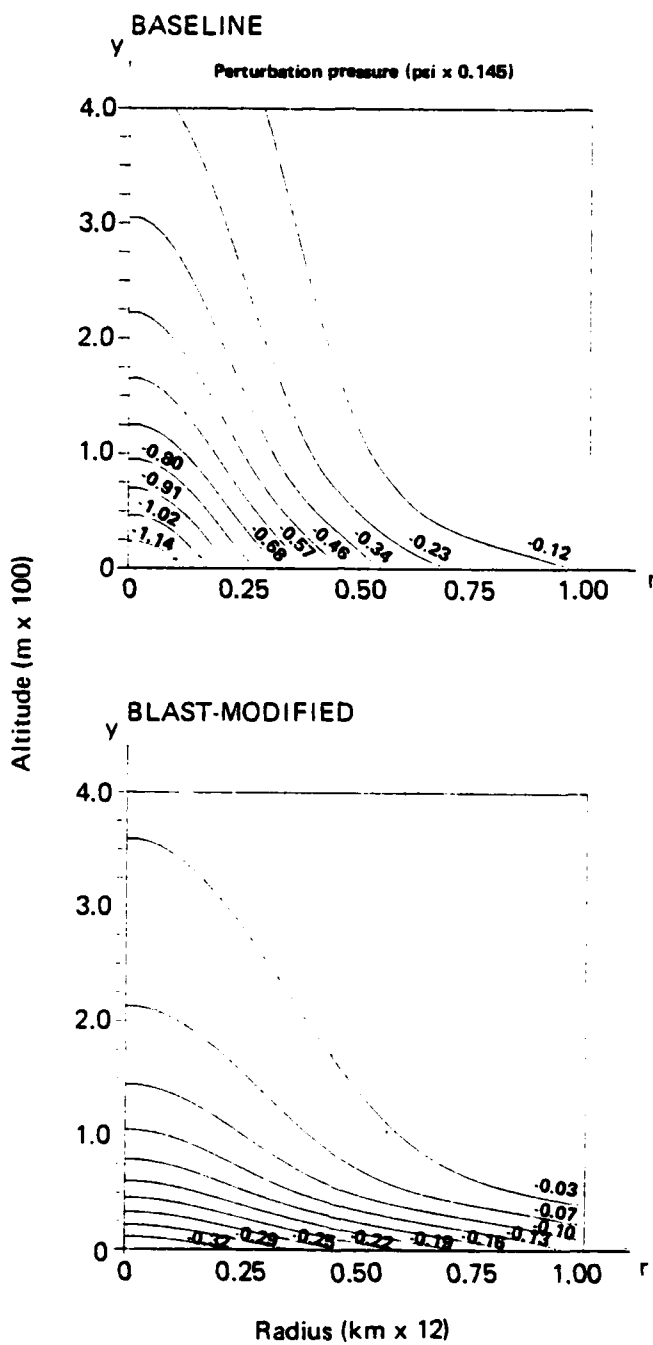


Fig. 40--Baseline and blast-modified pressure contours for city E

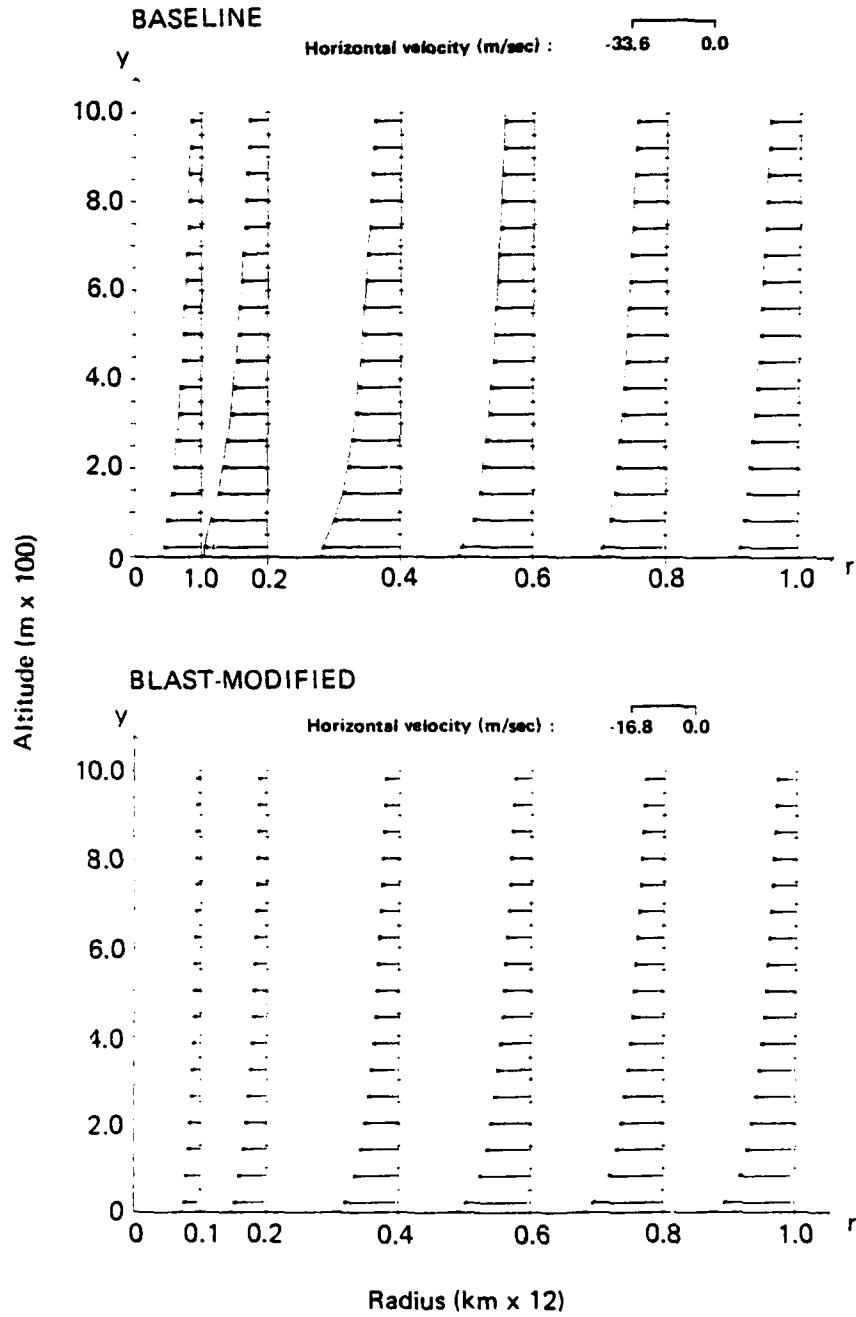


Fig. 41--Baseline and blast-modified radial velocity profiles for city E

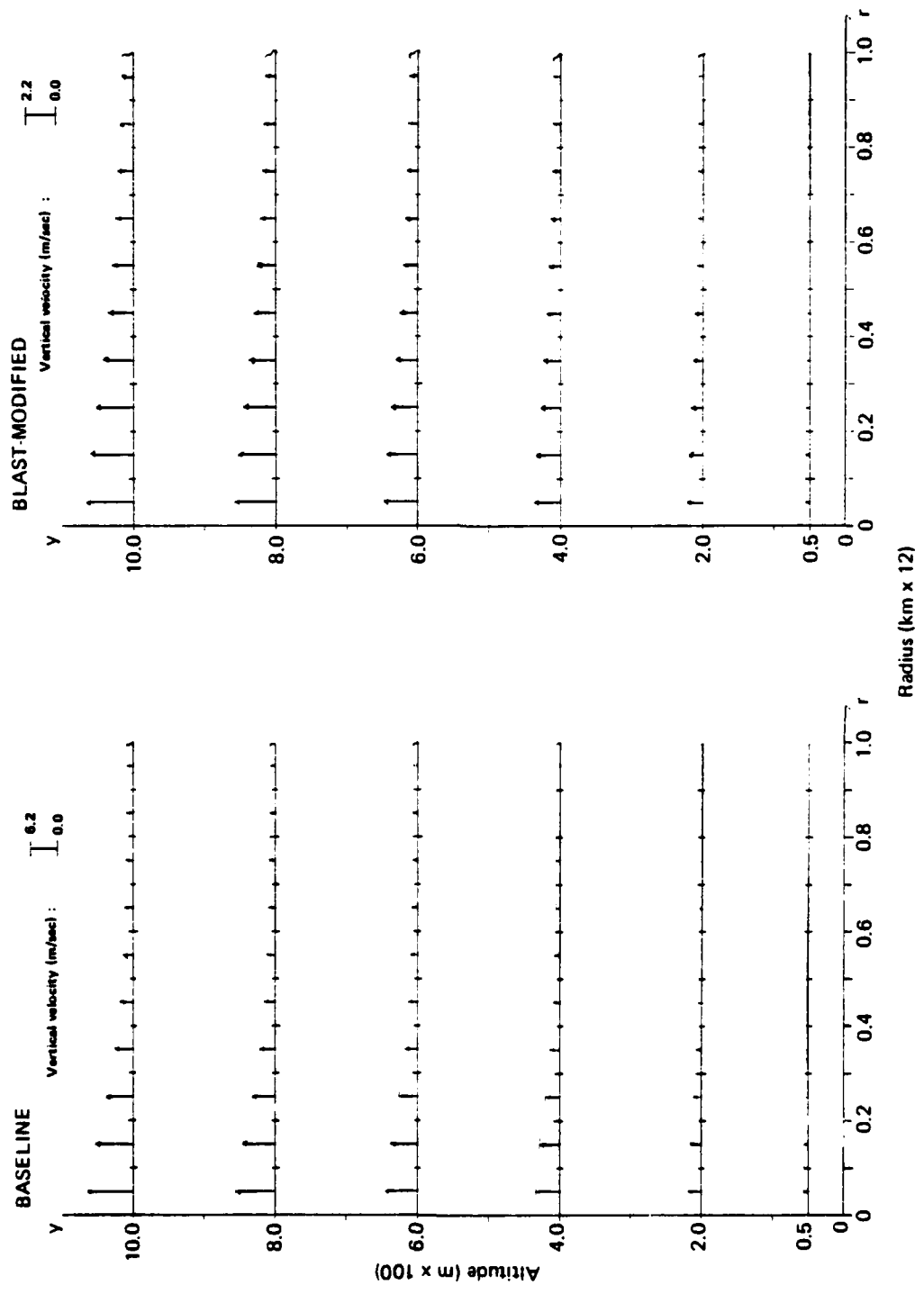


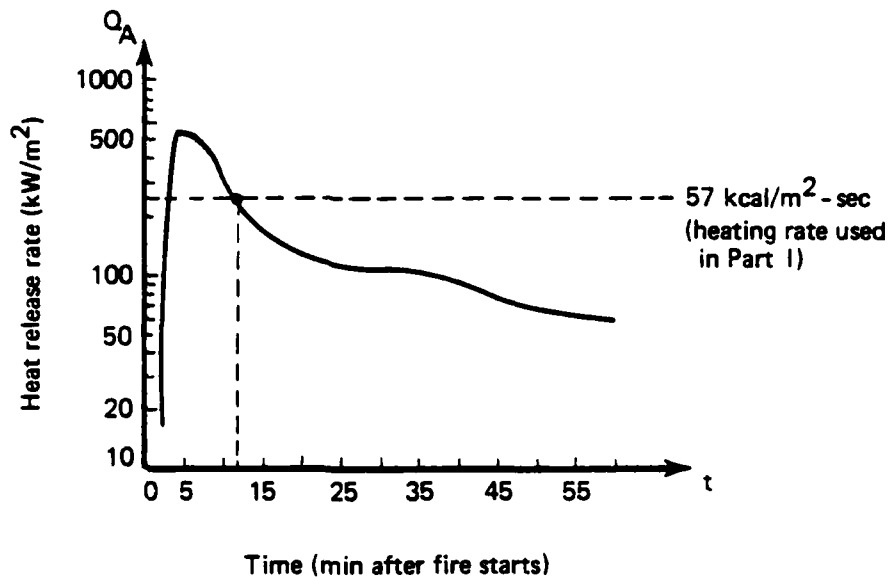
Fig. 42--Baseline and blast-modified vertical velocity profiles for city E

in the results presented in the corresponding plots is that temperatures, pressure drops, and velocities all increase with total loading, and are least for city W and greatest for city E. Further, as shown in Fig. 41, the baseline radial velocity for city E is not maximum at the fire periphery, as in the other cases (cf. Figs. 33 and 37), but rather about halfway between the periphery and the center. That difference presumably reflects the greater radial extent of the taller central-city and inner-belt regions in this case than in the others (cf. Figs. 27 through 29).

IV. SAMPLE TIME-DEPENDENT SIMULATION

This section extends the quasi-steady turning-region analysis of Part I to obtain time-dependent predictions. The multiple-fuel-bed Flambeau fire [Countryman, 1969; Palmer, 1981] treated in Part I is used as a sample case.

Figure 43 presents a typical time-history of the areal heat release in the largest Flambeau fires. During the period encompassing most of the burning (5 to 55 min after the fire starts), variations in the mean heat release rate occur on a time scale of many minutes. At any given point during that period, a steady-state analysis like that in Part I (Sec. IV) is justified as long as dynamic transients occur on a much shorter time scale of, say, a minute or less. As we now show, that criterion is satisfied at all times between 5 and 45 min. A time-dependent simulation is constructed by performing steady-state calculations at several points in that interval.



Source: Adapted from Fig. 3 of Palmer [1981]. Reprinted with permission from *Atmospheric Environment*, Vol. 15, Thomas Y. Palmer, "Large Fire Winds, Gases and Smoke," 1981, Pergamon Press, Ltd.

Fig. 43--Areal heat release time-history for Flambeau fire 760-12

In the steady-state Flambeau simulation, we used $R = 250$ m, $H = 20$ m, $QH \sim 50$ kcal/m²-sec, and hence had $U \sim 8$ m/sec [see Eqs. (2)]. For that case, the time scale R/U on which dynamic transients occurred was approximately 30 sec, satisfying the criterion for the flow to be quasi-steady. In the new, time-dependent analysis, we similarly use

$$\begin{aligned} R &= 250 \text{ m} , \\ H &= 20 \text{ m} , \end{aligned} \tag{10}$$

but vary QH with time as shown in Fig. 43. At any given time, the criterion for the flow to be quasi-steady is that $QH \gtrsim 25$ kcal/m²-sec. From Fig. 43, it is certainly satisfied for periods between 5 and 45 min, and possibly up to 55 min as well.

In the following simulation, we calculate quasi-steady flows at points 5 min apart. The specific values of QH used at those points are summarized in Table 10. As in the steady-state simulation of Part I, we also use

$$q(r, y) = \begin{cases} 1.6 & \text{for } 0 \leq y \leq 0.25 \\ 1.6 \left(\frac{4}{3} (1 - y) \right) & \text{for } 0.25 \leq y \leq 1.0 , \\ 0 & \text{for } y \geq 1.0 \end{cases}$$

$$k^{*-1} = 20 \text{ m} ,$$

$$\frac{g_1}{\rho_a} = \frac{k_1}{c_p \rho_a} = 8410 \text{ m}^2/\text{sec} \tag{11}$$

at each point. Thus, from Eqs. (2) and (3),

$$U = 8.41 \times F \quad \text{m/sec} ,$$

$$A = \frac{2.77}{F^2} ,$$

$$\sigma = \frac{0.022}{F} ,$$

$$M_1 = K_1 = \frac{4.0}{F} , \tag{12}$$

where

$$F = \frac{QH}{57 \text{ kcal/m}^2\text{-sec}} \tag{13}$$

and, from p. 6, $\delta = 9.04 \times 10^{-14} \times F^2$.

Figures 44 through 47 summarize the results of the time-dependent Flambeau simulation. The predicted variations with time of the temperature, pressure, and velocity fields are shown for the heat release described in Fig. 43. The solid curves represent the results of the quasi-steady analysis, which is considered appropriate for the period approximately 5 to 55 min after the fire starts.

Table 10

HEAT RELEASE SCALES USED IN TIME-DEPENDENT FLAMBEAU SIMULATION

Time (min)	QH (kcal/m ² -sec)
5	134
10	84
15	53
20	38
25	31
30	24
35	20
40	18
45	18
50	14
55	10

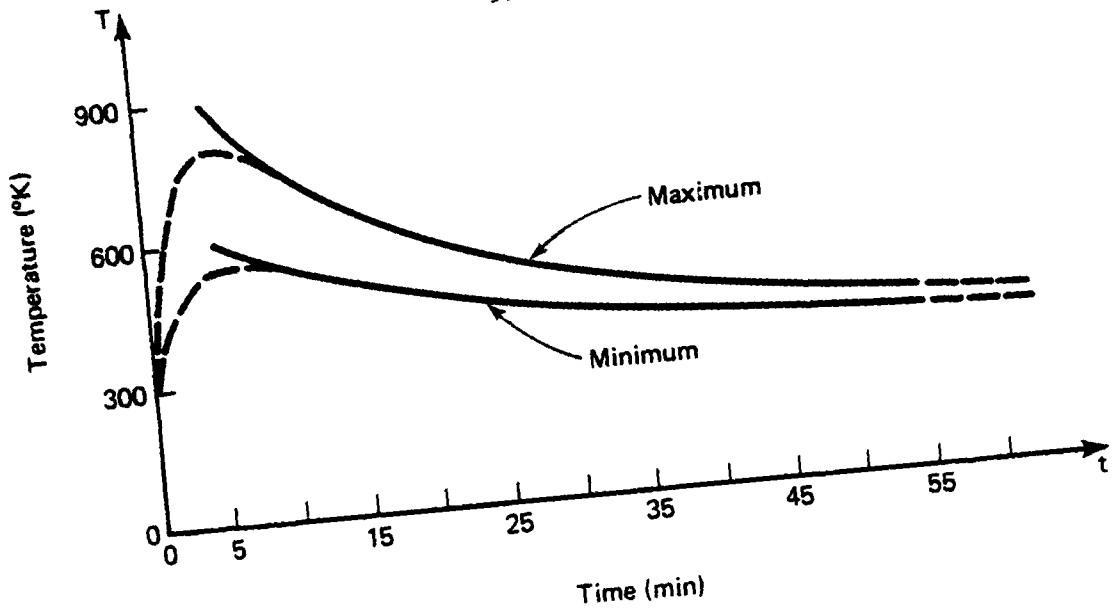


Fig. 44--Time-history of maximum and minimum combustion zone temperatures for sample Flambeau fire

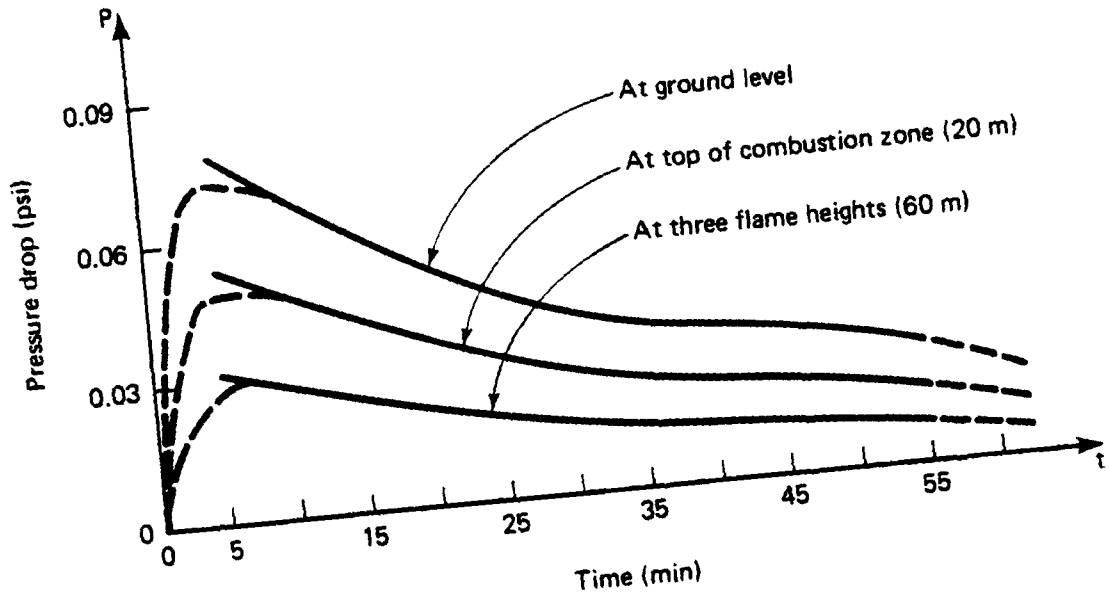


Fig. 45--Time-history of peripheral pressure drop for sample flambeau fire

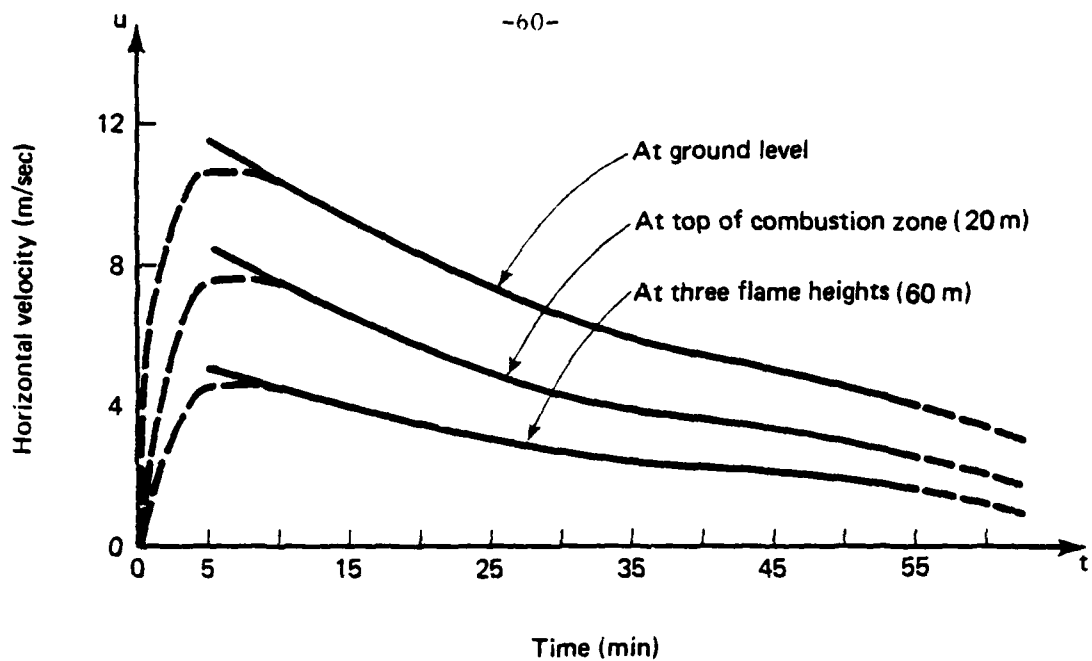


Fig. 46--Time-history of induced peripheral fire wind (radial velocity) for sample Flambeau fire

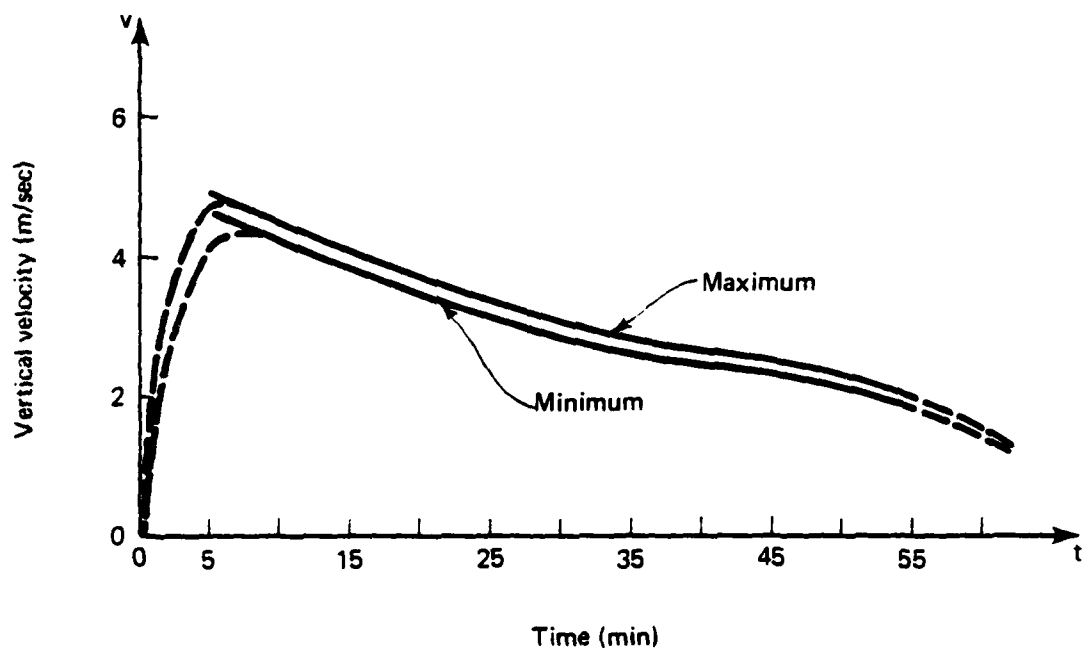


Fig. 47--Time-history of emerging column flow (vertical velocity at top of turning region) for sample Flambeau fire

During the first 5 min of burning, when the individual fires are coalescing to form a large-fire system, the heat release rate (Fig. 43) is changing on a time scale that is fairly fast compared with that for convection. The same is also true during the final phase of fire activity, i.e., the period later than 1 hr or so when the large fire begins to disintegrate and burn out. For those two periods, the quasi-steady analysis is thus inappropriate, and the dashed curves in Figs. 44 through 47 illustrate the expected results of a more rigorous analysis.

Figure 44 describes the predicted variation of the temperature field with time in the burning zone. The contours shown in Fig. 48 are representative of the temperature field at any given time. Maximum temperatures occur in the center of the fire just above the top of the fuel zone (height ≈ 4 m). Minimum temperatures occur on the periphery at the top of the burning region. As expected, the time-histories of those temperatures resemble the heat release time-history shown in Fig. 43. Temperatures are everywhere greatest about 5 min after the fire starts, when the heat release is greatest, and then decrease with time.

The corresponding pressure and velocity time-histories shown in Figs. 45 through 47 follow the same pattern. As illustrated in Figs. 49 and 50, pressure gradients and radial velocities at any given time are maximum at the fire periphery. At all times, they are greatest at ground level and decrease with altitude. Finally, Fig. 47 characterizes the variation with time of the vertical velocity emerging at the top of the computational region (10 flame heights, 200 m). As illustrated in Fig. 51, velocity profiles at that height are almost top-hat at all times. Maximum and minimum velocities, which occur above the fire center and on the periphery, respectively, are thus always quite similar.

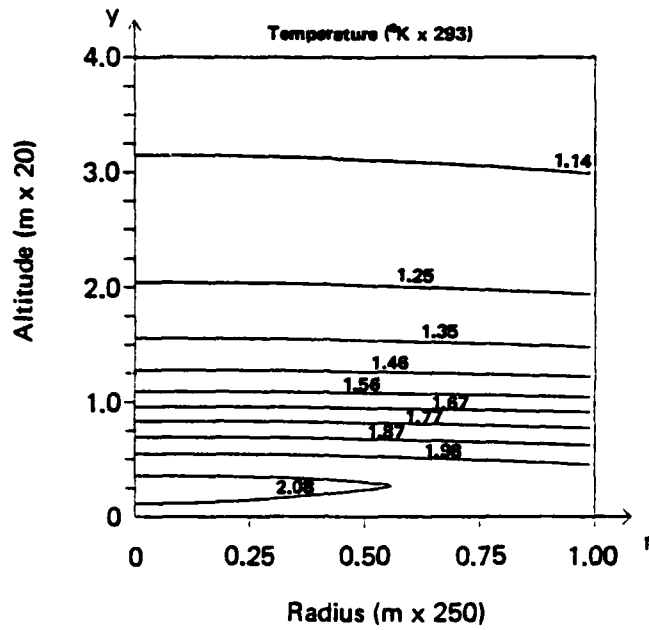
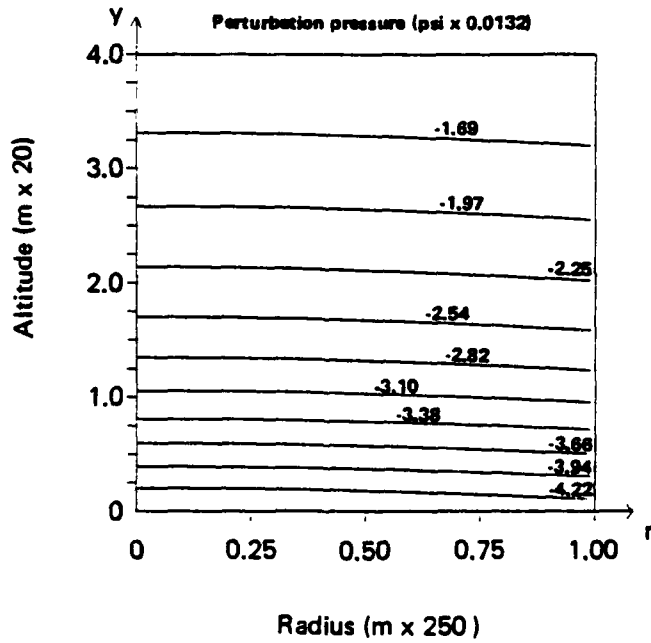


Fig. 48--Temperature contours after 15 min for sample Flambeau fire



Note: Perturbation pressure is $\delta P_g (P + A_y)$; see footnote † on p. 6.

Fig. 49--Pressure contours after 15 min for sample Flambeau fire

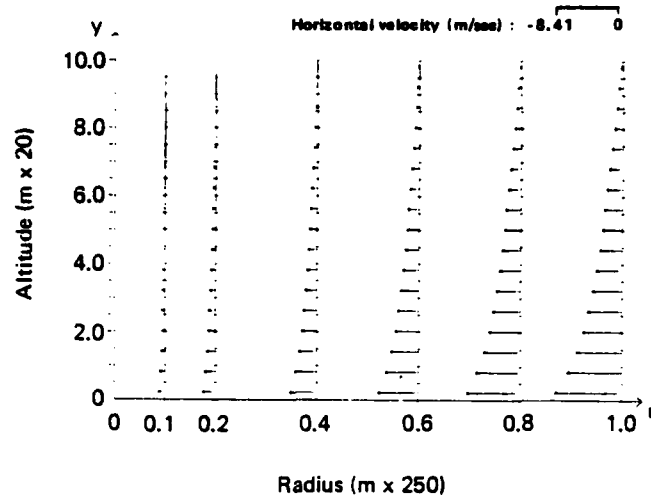


Fig. 50--Radial velocity profiles after 15 min for sample Flambeau fire

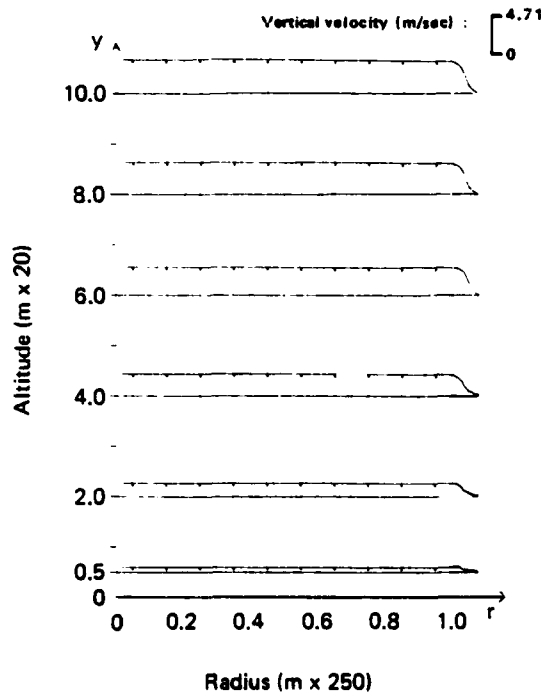


Fig. 51--Vertical velocity profiles after 15 min for sample Flambeau fire

V. DISCUSSION

In presenting calculations of the urban fire environment that may result from a nuclear weapon burst, our aim has been to describe the macroscopic features of a large area fire through analysis of the interactions that produce high temperatures and fire winds. Our approach has been to develop a model based on first principles. In contrast to asymptotic theories that extrapolate the properties of a free-convection plume, we consider the fire region directly. Despite our use of several simplifications and empiricisms, we expect that the theory yields a fine resolution of the hydrothermodynamics of a large area fire.

The analysis considers a quasi-steady axisymmetric fire in an urban area. A spatially dependent volume-heat-release function is used to describe the energy input from fires in a blast-disturbed urban fuel bed. Though the function can be defined to reflect varying fuel loads and distributions of burning, the results presented are based on sectionally uniform heating rates.

The formulation of the boundary value problem is unique in that analytic jump conditions are derived for the fire-column boundary. Such a formulation obviates the need for extensive far-field calculations and allows demonstration of the dependence of induced fire winds on the fire area and heat release rate. Improvements to the model could include the use of multiparameter models to describe the structure of local turbulence and hot-gas/smoke radiation (in place of the simple one-parameter formulations used here) as well as a heat release function dependent on the local thermodynamic and flow conditions.

The effects of the model parameters were explored in a sensitivity study. Varying fire areas, heat distributions and release rates, mixing coefficients, and radiation lengths were investigated. Qualitatively similar flows were observed for all parameter ranges, despite significant quantitative differences. The fire-induced velocities increased with both heat release and fire area. For small values, the

velocity increased almost linearly. For larger values of either fire area or heat release rate, the induced velocities increased more slowly. Presumably, the magnitudes are limited by compressibility effects through a slowing production of buoyancy.

Variations in the degree of turbulent mixing produced substantial changes in the velocity fields. As expected, large mixing coefficients tend to reduce the velocities at the fire periphery; smaller values produce higher velocities. In general, the mixing coefficients should reflect both the apparent surface roughness (urban structure) and the fire-generated turbulence. The present analysis used constant eddy diffusivities.

Fuel distributions and heat release rates were developed to explore the effects of varying city constructions and fuel densities on the fire environment in three model urban areas. Though the models provide only a simple representation of a city cross section, the results indicate the importance of modeling a particular fuel bed. More refined distributions based on city surveys could be used.

A fire radius of 12 km is assumed for each model urban area, since megaton-yield explosions would cause area fires of that magnitude. Each model area is characterized by a high-density city center, a lower density annulus representing a mixed residential/industrial construction, and an outer belt of low-density residential tracts. The heat release function reflected the sectionally variable fuel densities and loadings.

Computed results for each fire illustrate the dependence of the fire winds and temperature levels on the urban geography. The most severe fires occur in the higher density cities, though even the lower density constructions support high temperatures and velocities. Application of those results to definition of shelter hardness (thermal) would imply different criteria for each city.

An additional set of numerical experiments considered each model area with a reduced level of burning in the center region. A slow-burning debris field created by extensive blast damage may foster such a heat release distribution. In general, peak velocities were only

slightly reduced, though temperature changes were substantial. Temperature levels in the central region were markedly reduced from those in the full-burning outer annulus. It is reasonable to expect that the fire winds would fan the slow-burning central area, creating a more severe environment.

The analysis and results presented here describe the environment of a large area fire at fixed times. Past experience indicates that such fires develop, peak, and ebb over a period of several hours. The methods developed here can be used successively to estimate the environment at different times in the fire evolution. The high velocities and temperatures predicted would encourage a rapid internal fire spread, which may considerably hasten burnout of the city. Improved estimates would result from use of a responsive heat release function that depends on the local gas dynamic state. A time-dependent calculation that includes the high-resolution analysis presented here could provide a complete map of the evolution and ebb of urban fires. Synthesis of the present analysis with a large hydro-code may provide a numerically efficient model for calculating the mesoscale flow field of a large area fire.

Appendix

PREDICTION-ALGORITHM DOCUMENTATION

This appendix documents the computer algorithm used to generate the predictions and other results presented in Secs. II through IV. The algorithm yields numerical solutions to the turning-region boundary value problem derived in Part I.

The turning-region problem is defined by the following balances:

$$\frac{\partial}{\partial r} (r\rho u) + \frac{\partial}{\partial y} (r\rho v) = 0 ,$$

$$\rho \left(u \frac{\partial u}{\partial r} + v \frac{\partial u}{\partial y} \right) = - \frac{\partial P}{\partial r} + M_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u}{\partial r} \right) - \frac{u}{r^2} \right) ,$$

$$\frac{\partial P}{\partial y} + A\rho = 0 ,$$

$$\rho \left(u \frac{\partial T}{\partial r} + v \frac{\partial T}{\partial y} \right) = q(r, y) - \sigma(T^4 - 1) + K_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) \right) ,$$

$$\rho T = 1 , \tag{A.1}$$

subject to

$$v = 0 \quad \text{on } y = 0 , \tag{A.2a}$$

$$u = \frac{\partial T}{\partial r} = 0 \quad \text{on } r = 0 , \tag{A.2b}$$

$$P + Ay_{\max} = 0 \quad \text{on } y = y_{\max} , \tag{A.2c}$$

$$M_1 \frac{\partial u}{\partial r} = \left(\frac{u}{T} \right)^2 (T - 1) + (P + Ay) , \quad K_1 \frac{\partial T}{\partial r} = \frac{u}{T} (T - 1) \tag{A.2d}$$

on $r = 1$.

The flow chart in Fig. A.1 summarizes the solution algorithm. As indicated in the figure (and discussed in Part I, pp. 16-17), the algorithm uses an iterative "shooting" method [Keller, 1968] to solve the overall boundary value problem. Each shoot employs a nonlinear Crank-Nicolson finite difference scheme.

INPUTS AND OUTPUTS

The following is an ordered list of the FORTRAN inputs required for automated algorithm usage:

a. System parameters:

A	A
SIGMA	σ
XNUIN	M_1
XKIN	K_1

b. Mesh descriptors:

YMAX	y_{\max}
M	Number of finite difference cells in the radial direction. [†]
N	Number of finite difference cells in the vertical direction. [†]

c. Iteration parameters:

ETLINE	Maximum absolute error allowed in solving discretized Crank-Nicolson equations on any line of constant y .
ETOLP	Maximum absolute error allowed in P on $y = y_{\max}$.
MAXITL	Maximum number of Newton iterations allowed in solving discretized equations on any line of constant y .
MAXITP	Maximum number of complete-shoot iterations allowed in solving the complete boundary value problem.

† In the present coding, cell widths are taken to be uniform, the common width being denoted by Δr . For $0 \leq y \leq 5$, cell heights Δy are also taken to be Δr ; for $5 \leq y \leq y_{\max}$, they are taken to be $2 \Delta r$.

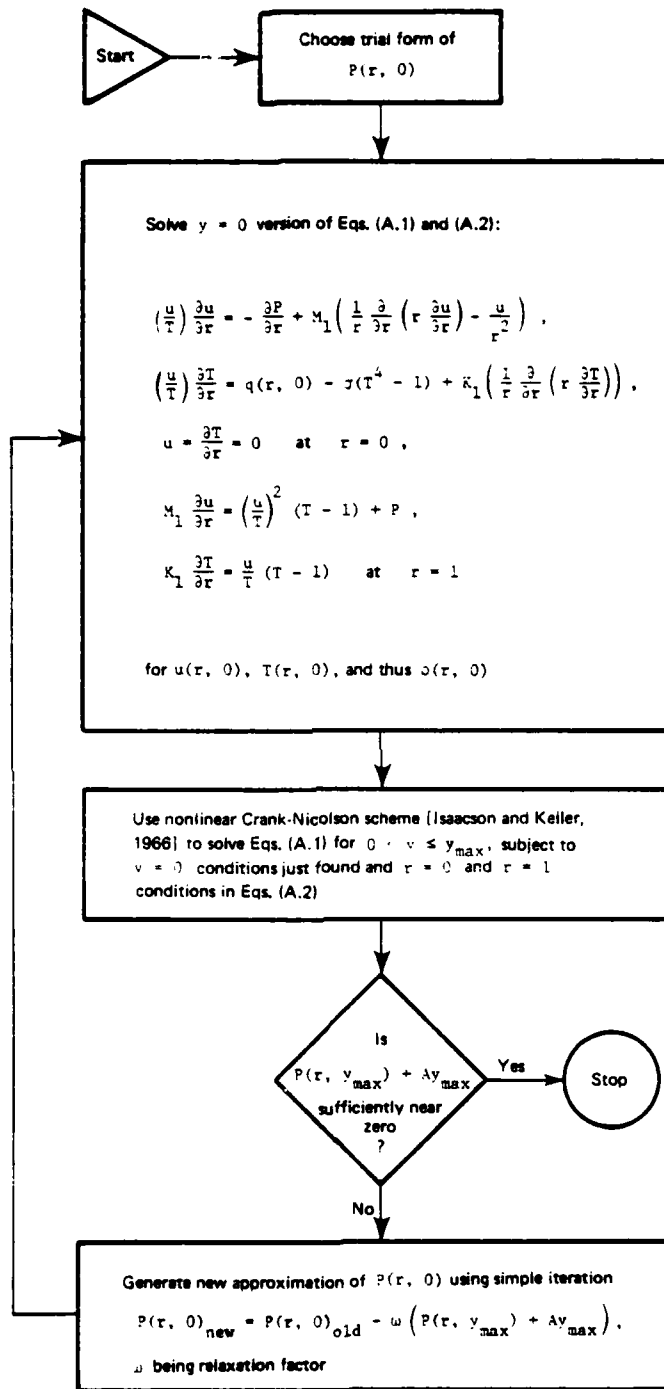


Fig. A.1--Flow chart of prediction algorithm for turning-region boundary value problem

d. Program control parameters:

ALPHA $(Qq)_{\max}$, maximum value of Qq in present coding.
BETA, BLOSS Parameters available for either system description or program control. (These parameters are currently unused, so they are set equal to zero.)
IGUESS Parameter that controls the initial choices of first-shoot values along $y = 0$.
IGUESS = 1: Discretized versions of $P(r, 0)$, $T(r, 0)$, and $u(r, 0)$ must be supplied as inputs.
IGUESS = 2: "Standard" initial choices are made automatically.
IGUESS = 3: Discretized versions of $P(r, 0)$, $T(r, 0)$, and $u(r, 0)$ from the last algorithm run are used automatically.

The "standard" initial guesses in the present coding represent the ground-level pressure, temperature, and radial velocity profiles predicted in Part I (pp. 13-16) for the case of a weakly heated flow:

$$\begin{aligned} P(r, 0) &= - (Qq)_{\max} \left(\frac{A}{4\sigma} \right) , \\ T(r, 0) &= 1.0 + (Qq)_{\max} \left(\frac{1}{4\sigma} \right) , \\ u(r, 0) &= - (Qq)_{\max} \left(\frac{A}{4\sigma M_1} \right) r . \end{aligned} \tag{A.3}$$

Finally, the functional form of $q(r, y)$ must be prescribed. Currently, it is given in the form used in our analysis:

$$q(r, y) = \left\{ \begin{array}{ll} (Qq)_{\max} & \text{for } 0 \leq y \leq 0.25 \\ (Qq)_{\max} \left(\frac{4}{3} (1 - y) \right) & \text{for } 0.25 \leq y \leq 1.0 \\ 0 & \text{for } y \geq 1.0 \end{array} \right\} . \quad (\text{A.4})$$

Changes require reprogramming in subroutine BCFUNC. As a user option, the dissipation coefficients M_1 and K_1 may also be made to depend on r and y through reprogramming, also in BCFUNC. Currently, those coefficients are simply taken as constants.

A sample input data stream is

A = 0.217 , SIGMA = 0.110 , XNUIN = XKIN = 0.200 ;

YMAX = 10 , M = 40 , N = 300 ;

ETLINE = 0.0003 , ETOLP = 0.001 , MAXITL = 30 , MAXITP = 9 ;

ALPHA = 1.6 , BETA = 0.0 , BLOSS = 0.0 , I GUESS = 3 . (\text{A.5})

The corresponding output is listed in Table A.1. All output data are automatically dumped onto a disk file and used in constructing the vector, contour, and profile plots presented in Secs. II through IV.

PROGRAM .LOW

Figure A.1 outlines the overall flow of the computer algorithm. Inputs, outputs, and the basic shooting iteration are controlled by the main program. Figure A.2 presents a flow chart of that program. In subroutine BCFUNC, functional forms that may be varied [e.g., $q(r, y)$] are specified; and in subroutine SPRINT, final output data are printed. Subroutine MSWEEP performs most of the computations-- i.e., those providing a one-shoot finite difference solution to Eqs. (A.1) for given data along $y = 0$, $r = 0$, and $r = 1$. The flow of that subroutine is shown in Figs. A.4 and A.5 below (pp. 80-81); the underlying numerical analysis is detailed in the accompanying

Table A.1
SAMPLE CODE OUTPUT

FIRE: SOLUTION OF COMBUSTION LAYER BOUNDARY VALUE PROBLEM

SYSTEM PARAMETERS: A = 0.217 SIGMA = 0.110 NU = 0.200 K = 0.200

HEM: M = 40 NR = 0.025 N = 300 DY = 0.025

TITRATION PARAMETERS: FTIME = 0.000100 MAXITL = %0 FTOLP = 0.001000 MAXITP = 9

INLET PARAMETERS: ALPHA = 1.000 BETA = 0.000 MUOSS = 0.000

FOR J = 1 (Y = 0.000):

I	R	U	V	T	RHO	P	PR	MIXR	K*TR	Q(T)
1	0.000	0.000	0.000	1.938	0.516	-0.183	0.000	0.000	0.000	0.158
2	0.025	-0.019	0.000	1.938	0.516	-0.183	0.000	0.000	0.000	0.158
3	0.050	-0.039	0.000	1.938	0.516	-0.183	0.000	0.000	0.000	0.158
4	0.075	-0.057	0.000	1.938	0.516	-0.183	0.000	0.000	-0.320	0.158
5	0.100	-0.074	0.000	1.937	0.516	-0.183	0.000	0.000	0.000	0.161
6	0.125	-0.089	0.000	1.936	0.517	-0.182	0.000	0.000	-0.320	0.165
7	0.150	-0.113	0.000	1.934	0.517	-0.182	0.000	0.000	-0.320	0.171
8	0.175	-0.132	0.000	1.931	0.518	-0.181	0.000	0.000	-0.320	0.177
9	0.200	-0.150	0.000	1.928	0.518	-0.180	0.000	0.000	-0.320	0.181
10	0.225	-0.168	0.000	1.926	0.519	-0.179	0.000	0.000	-0.320	0.186
11	0.250	-0.186	0.000	1.925	0.520	-0.178	0.000	0.000	-0.320	0.190
12	0.275	-0.204	0.000	1.923	0.521	-0.177	0.000	0.000	-0.320	0.196
13	0.300	-0.221	0.000	1.921	0.522	-0.177	0.000	0.000	-0.320	0.201
14	0.325	-0.238	0.000	1.919	0.523	-0.176	0.000	0.000	-0.320	0.206
15	0.350	-0.255	0.000	1.917	0.524	-0.176	0.000	0.000	-0.320	0.211
16	0.375	-0.272	0.000	1.915	0.524	-0.175	0.000	0.000	-0.320	0.216
17	0.400	-0.288	0.000	1.913	0.525	-0.175	0.000	0.000	-0.320	0.221
18	0.425	-0.304	0.000	1.911	0.525	-0.174	0.000	0.000	-0.320	0.226
19	0.450	-0.319	0.000	1.909	0.526	-0.174	0.000	0.000	-0.320	0.231
20	0.475	-0.334	0.000	1.907	0.526	-0.169	0.000	0.000	0.000	0.237
21	0.500	-0.349	0.000	1.905	0.527	-0.165	0.000	0.000	-0.320	0.243
22	0.525	-0.363	0.000	1.903	0.528	-0.164	0.000	0.000	0.000	0.249
23	0.550	-0.377	0.000	1.901	0.528	-0.162	0.000	0.000	0.000	0.255
24	0.575	-0.390	0.000	1.899	0.529	-0.160	0.000	0.000	-0.320	0.261
25	0.600	-0.404	0.000	1.897	0.529	-0.158	0.000	0.000	0.000	0.267
26	0.625	-0.416	0.000	1.895	0.529	-0.156	0.000	0.000	-0.320	0.273
27	0.650	-0.428	0.000	1.893	0.529	-0.154	0.000	0.000	0.000	0.279
28	0.675	-0.440	0.000	1.891	0.529	-0.152	0.000	0.000	-0.320	0.285
29	0.700	-0.452	0.000	1.889	0.529	-0.150	0.000	0.000	0.000	0.291
30	0.725	-0.464	0.000	1.887	0.529	-0.148	0.000	0.000	-0.320	0.297
31	0.750	-0.477	0.000	1.885	0.529	-0.146	0.000	0.000	0.000	0.303
32	0.775	-0.489	0.000	1.883	0.529	-0.144	0.000	0.000	-0.320	0.309
33	0.800	-0.502	0.000	1.881	0.529	-0.141	0.000	0.000	0.000	0.315
34	0.825	-0.514	0.000	1.879	0.529	-0.139	0.000	0.000	-0.320	0.321
35	0.850	-0.510	0.000	1.877	0.529	-0.137	0.000	0.000	0.000	0.327
36	0.875	-0.519	0.000	1.875	0.529	-0.134	0.000	0.000	-0.320	0.333
37	0.900	-0.527	0.000	1.873	0.529	-0.131	0.000	0.000	0.000	0.339
38	0.925	-0.534	0.000	1.871	0.529	-0.129	0.000	0.000	-0.320	0.345
39	0.950	-0.541	0.000	1.869	0.529	-0.126	0.000	0.000	0.000	0.351
40	0.975	-0.548	0.000	1.867	0.529	-0.123	0.000	0.000	-0.320	0.357
41	1.000	-0.554	0.000	1.865	0.529	-0.120	0.000	0.000	0.000	0.363

Table A.1--Continued

FOR J = P . (V = 0.025):

I	R	II	V	T	RHO	P	PR	MIARR	KSTR	Q(T)
1	0.000	0.000	0.036	1.941	0.515	-0.180	0.000	0.000	0.000	0.154
2	0.025	-0.017	0.036	1.941	0.514	-0.180	0.000	0.014	-0.099	0.154
3	0.050	-0.033	0.036	1.940	0.515	-0.180	0.000	0.014	-0.079	0.155
4	0.075	-0.050	0.036	1.940	0.516	-0.180	0.020	0.045	-0.078	0.156
5	0.100	-0.067	0.035	1.939	0.516	-0.179	0.000	0.041	-0.078	0.159
6	0.125	-0.083	0.035	1.938	0.516	-0.179	0.000	0.015	-0.080	0.162
7	0.150	-0.099	0.035	1.936	0.516	-0.179	0.020	0.045	-0.082	0.162
8	0.175	-0.114	0.035	1.935	0.517	-0.178	0.040	0.069	-0.085	0.172
9	0.200	-0.131	0.035	1.933	0.517	-0.177	0.040	0.067	-0.089	0.178
10	0.225	-0.147	0.035	1.933	0.518	-0.176	0.020	0.043	-0.092	0.185
11	0.250	-0.164	0.034	1.929	0.518	-0.176	0.020	0.046	-0.096	0.192
12	0.275	-0.178	0.034	1.926	0.519	-0.175	0.000	0.070	-0.101	0.201
13	0.300	-0.190	0.034	1.923	0.520	-0.174	0.060	0.092	-0.106	0.211
14	0.325	-0.209	0.034	1.920	0.521	-0.172	0.060	0.090	-0.112	0.221
15	0.350	-0.224	0.033	1.916	0.522	-0.171	0.080	0.067	-0.119	0.233
16	0.375	-0.238	0.033	1.912	0.523	-0.170	0.080	0.068	-0.125	0.246
17	0.400	-0.252	0.032	1.907	0.524	-0.169	0.060	0.091	-0.133	0.260
18	0.425	-0.264	0.032	1.903	0.526	-0.167	0.060	0.089	-0.141	0.275
19	0.450	-0.280	0.032	1.897	0.527	-0.166	0.080	0.109	-0.150	0.291
20	0.475	-0.293	0.032	1.892	0.529	-0.164	0.060	0.086	-0.159	0.309
21	0.500	-0.306	0.031	1.895	0.530	-0.162	0.060	0.086	-0.170	0.327
22	0.525	-0.319	0.031	1.879	0.532	-0.161	0.080	0.086	-0.180	0.346
23	0.550	-0.332	0.031	1.871	0.534	-0.159	0.079	0.104	-0.204	0.393
24	0.575	-0.348	0.030	1.863	0.537	-0.157	0.079	0.102	-0.216	0.417
25	0.600	-0.356	0.030	1.855	0.539	-0.156	0.079	0.100	-0.229	0.443
26	0.625	-0.368	0.030	1.845	0.542	-0.154	0.079	0.100	-0.243	0.471
27	0.650	-0.379	0.029	1.835	0.545	-0.152	0.079	0.099	-0.258	0.501
28	0.675	-0.390	0.029	1.825	0.548	-0.150	0.079	0.097	-0.273	0.531
29	0.700	-0.400	0.029	1.813	0.552	-0.148	0.079	0.095	-0.287	0.564
30	0.725	-0.411	0.029	1.801	0.555	-0.146	0.079	0.094	-0.305	0.598
31	0.750	-0.421	0.029	1.787	0.559	-0.144	0.079	0.092	-0.320	0.634
32	0.775	-0.431	0.028	1.773	0.564	-0.142	0.099	0.111	-0.337	0.671
33	0.800	-0.440	0.028	1.758	0.569	-0.139	0.099	0.108	-0.353	0.709
34	0.825	-0.449	0.028	1.742	0.574	-0.137	0.079	0.085	-0.370	0.749
35	0.850	-0.458	0.028	1.724	0.580	-0.135	0.119	0.122	-0.386	0.791
36	0.875	-0.467	0.028	1.706	0.586	-0.132	0.098	0.098	-0.402	0.833
37	0.900	-0.475	0.028	1.686	0.594	-0.129	0.098	0.095	-0.417	0.877
38	0.925	-0.482	0.028	1.665	0.601	-0.127	0.118	0.113	-0.431	0.921
39	0.950	-0.490	0.028	1.643	0.609	-0.124	0.118	0.109	-0.445	0.966
40	0.975	-0.497	0.028	1.619	0.614	-0.121	0.118	0.109	-0.445	1.012
41	1.000	-0.504	0.028	1.594	0.627	-0.118	0.118	0.109	-0.445	1.012

Table A.1--Continued

FOR J = 101 (V = 10.000):

I	R	U	V	Y	RHO	P	PR	MUHR	KATR	Q(I)
1	0.000	0.000	4.035	1.000	0.992	0.000	0.000	0.000	0.000	-0.003
2	0.025	-0.002	4.034	1.000	0.992	0.000	-0.003	-0.002	0.000	-0.003
3	0.050	-0.005	4.033	1.000	0.992	0.000	-0.007	-0.005	-0.001	-0.003
4	0.075	-0.007	4.015	1.000	0.992	0.000	0.000	0.017	-0.002	-0.003
5	0.100	-0.010	4.785	1.000	0.992	0.001	0.005	0.012	-0.002	-0.003
6	0.125	-0.012	4.771	1.000	0.992	0.000	-0.019	-0.013	-0.001	-0.003
7	0.150	-0.014	4.754	1.000	0.992	0.000	-0.003	0.010	-0.002	-0.003
8	0.175	-0.017	4.712	1.000	0.993	0.000	-0.013	0.020	-0.001	-0.003
9	0.200	-0.019	4.657	1.000	0.993	0.001	0.013	0.024	-0.002	-0.003
10	0.225	-0.021	4.617	1.007	0.993	0.001	-0.014	0.000	-0.001	-0.003
11	0.250	-0.023	4.508	1.007	0.993	0.000	-0.017	-0.001	-0.001	-0.003
12	0.275	-0.025	4.547	1.007	0.993	0.000	-0.001	0.019	-0.001	-0.003
13	0.300	-0.026	4.403	1.007	0.993	0.000	0.016	0.037	-0.001	-0.003
14	0.325	-0.030	4.403	1.007	0.993	0.001	0.013	0.033	-0.001	-0.003
15	0.350	-0.031	4.140	1.007	0.993	0.000	-0.010	0.010	-0.001	-0.003
16	0.375	-0.033	4.200	1.007	0.993	0.000	-0.013	0.009	-0.001	-0.003
17	0.400	-0.035	4.226	1.007	0.993	0.000	0.004	0.024	-0.001	-0.003
18	0.425	-0.037	4.154	1.007	0.993	0.000	-0.001	0.025	-0.001	-0.003
19	0.450	-0.039	4.080	1.007	0.993	0.000	-0.016	0.023	-0.000	-0.003
20	0.475	-0.040	3.997	1.007	0.993	0.000	-0.007	0.041	-0.000	-0.003
21	0.500	-0.042	3.916	1.007	0.993	0.001	-0.009	0.018	-0.001	-0.003
22	0.525	-0.043	3.848	1.007	0.993	0.000	-0.009	0.016	0.000	-0.003
23	0.550	-0.044	3.770	1.007	0.993	0.000	-0.004	0.035	0.000	-0.003
24	0.575	-0.046	3.683	1.007	0.993	0.000	0.006	0.032	0.000	-0.003
25	0.600	-0.047	3.597	1.007	0.993	0.000	0.004	0.029	0.000	-0.003
26	0.625	-0.048	3.515	1.007	0.994	0.000	-0.002	0.027	0.000	-0.003
27	0.650	-0.049	3.435	1.006	0.994	0.000	-0.001	0.024	0.001	-0.003
28	0.675	-0.050	3.357	1.006	0.994	0.000	-0.003	0.022	0.000	-0.003
29	0.700	-0.051	3.283	1.006	0.994	0.000	-0.005	0.020	0.001	-0.003
30	0.725	-0.051	3.217	1.006	0.994	0.000	-0.007	0.018	0.000	-0.003
31	0.750	-0.052	3.145	1.006	0.994	0.000	-0.010	0.015	0.001	-0.003
32	0.775	-0.053	3.069	1.006	0.994	0.000	0.008	0.033	0.000	-0.003
33	0.800	-0.054	2.985	1.006	0.994	0.000	0.005	0.030	0.001	-0.003
34	0.825	-0.054	2.917	1.006	0.994	0.000	-0.017	0.006	0.001	-0.003
35	0.850	-0.054	2.854	1.006	0.994	-0.001	0.000	0.024	0.001	-0.003
36	0.875	-0.055	2.772	1.006	0.994	0.000	-0.006	0.041	0.001	-0.003
37	0.900	-0.055	2.694	1.006	0.994	0.000	-0.006	0.016	0.000	-0.003
38	0.925	-0.055	2.636	1.006	0.994	0.000	-0.009	0.013	0.002	-0.003
39	0.950	-0.055	2.571	1.006	0.994	0.000	0.007	0.029	0.001	-0.003
40	0.975	-0.055	2.500	1.006	0.994	0.000	0.003	0.075	0.000	-0.003
41	1.000	-0.055	2.436	1.006	0.994	0.000	0.001	0.025	0.000	-0.003

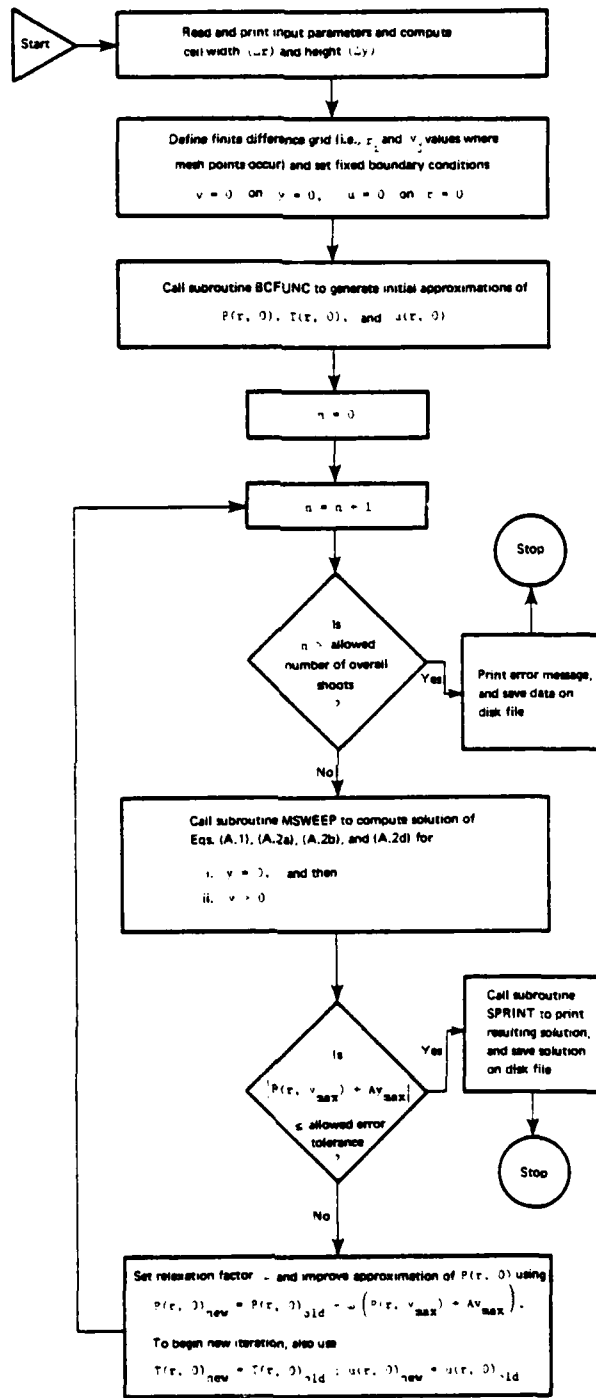


Fig. A.2--Flow chart of main program

text. Complete FORTRAN listings of the main program and all sub-routines are presented at the end of this appendix.

FINITE DIFFERENCE ALGORITHM AND SUBROUTINE MSWEEP

The finite difference scheme underlying the computer algorithm is based on the rectangular grid and "stencils" in Fig. A.3. The grid has uniform cell widths ($\Delta r = 1/M$) and piecewise uniform heights ($\Delta y = \Delta r$ for $0 \leq y \leq 5$, $y = 2 \Delta r$ for $y > 5$). Mesh points are thus located at (r_i, y_j) points with

$$r_i = (i - 1) \Delta r, \quad 1 \leq i \leq M + 1,$$

$$y_j = \begin{cases} (j - 1) \Delta r, & 1 \leq j \leq 5M + 1 \\ 5 + (j - 5M - 1)(2 \Delta r), & 5M + 2 \leq j \leq N + 1 \end{cases}. \quad (A.6)$$

Dependent variables evaluated at those points are given corresponding (i, j) suffices. For example, $r_1 = 0$ (and not Δr), $y_{N+1} = y_{\max}$, and $u_{1,N+1} = u(0, y_{\max})$. The accurate approximation of $\{u_{i,j}, v_{i,j}, P_{i,j}, T_{i,j}, \rho_{i,j}\}_{1 \leq i \leq M+1, 1 \leq j \leq N+1}$ constitutes a numerical solution of the problem posed by Eqs. (A.1) and (A.2). The stencils in Fig. A.3, explained below (pp. 82-83), indicate the types of differencing used in the algorithm.

Along the $y = 0$ line (where $j = 1$), $v \equiv 0$. As shown in Fig. A.2, a guess at P along that line [actually, the $\{P_{i1}\}_{i=1}^{M+1}$ values] is also made before the MSWEEP subroutine is called. As indicated in the first large block of Fig. A.1, Eqs. (A.1) and (A.2) then reduce along the $y = 0$ line to the following ordinary-differential-equation boundary value problem for u and T alone (with $\rho = 1/T$):

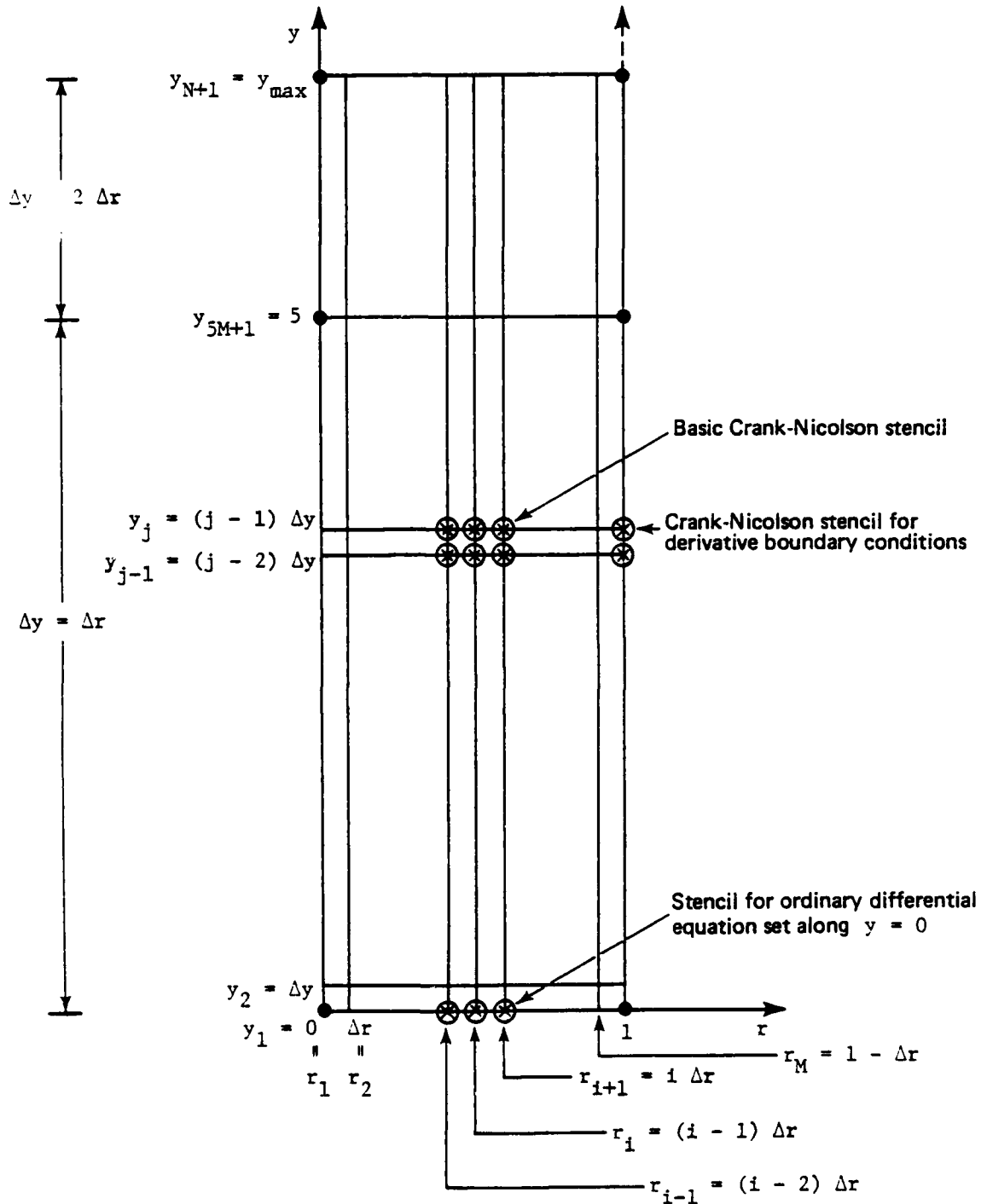


Fig. A.3--Finite difference grid and stencils for numerical solution of turning-region problem

$$\left(\frac{u}{T}\right) \frac{\partial u}{\partial r} = -\frac{\partial P}{\partial r} + M_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u}{\partial r} \right) - \frac{u}{r^2} \right),$$

$$\left(\frac{u}{T}\right) \frac{\partial T}{\partial r} = q(r, 0) - \sigma(T^4 - 1) + K_1 \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) \right),$$

$$u = \frac{\partial T}{\partial r} = 0 \quad \text{at } r = 0,$$

$$M_1 \frac{\partial u}{\partial r} = \left(\frac{u}{T}\right)^2 (T - 1) + P, \quad K_1 \frac{\partial T}{\partial r} = \frac{u}{T} (T - 1)$$

$$\text{at } r = 1. \quad (\text{A.7})$$

The standard finite-difference solution for this problem involves the solution of

$$\vec{F}(\vec{x}) = \vec{0}, \quad (\text{A.8})$$

where

$$\vec{x} \equiv \begin{bmatrix} \tilde{x}_1 \\ \tilde{x}_2 \\ \tilde{x}_3 \\ \tilde{x}_4 \\ \vdots \\ \tilde{x}_{2M-3} \\ \tilde{x}_{2M-2} \\ \tilde{x}_{2M-1} \\ \tilde{x}_{2M} \end{bmatrix} = \begin{bmatrix} u_{2,1} \\ T_{2,1} \\ u_{3,1} \\ T_{3,1} \\ \vdots \\ u_{M,1} \\ T_{M,1} \\ u_{M+1,1} \\ T_{M+1,1} \end{bmatrix} \quad (\text{A.9})$$

and the components of \vec{F} are the finite difference equations obtained from the discretization of Eqs. (A.7). The Crank-Nicolson solution of Eqs. (A.1)--subject to Eqs. (A.2b) and (A.2d), and the solution of Eqs. (A.7) [with $P(r, 0)$ given]--also involves the solution of Eq. (A.8) for successive j , $j = 2, 3, 4, \dots, N, N + 1$. In that case, the components of \vec{F} are slightly different from those for $j = 1$. For each fixed $j(\geq 2)$, \vec{x} is redefined [cf. Eq. (A.9)] as

$$\vec{x} \equiv \begin{pmatrix} \tilde{x}_1 \\ \tilde{x}_2 \\ \tilde{x}_3 \\ \tilde{x}_4 \\ \vdots \\ \tilde{x}_{2M-3} \\ \tilde{x}_{2M-2} \\ \tilde{x}_{2M-1} \\ \tilde{x}_{2M} \end{pmatrix} = \begin{pmatrix} u_{2,j} \\ T_{2,j} \\ u_{3,j} \\ T_{3,j} \\ \vdots \\ u_{M,j} \\ T_{M,j} \\ u_{M+1,j} \\ T_{M+1,j} \end{pmatrix} \quad (A.10)$$

The overall flow in subroutine MSWEEP is thus as outlined in Fig. A.4, Newton's method (see Fig. A.5) being used in all cases to solve the component equations of Eq. (A.8) because they are nonlinear.

To fully describe subroutine MSWEEP, we must define the following processes:

1. The specific finite-difference schemes used to reduce Eqs. (A.7) to Eq. (A.8) for $j = 1$ ($y = 0$), and Eqs. (A.1), (A.2b), and (A.2d) to Eq. (A.8) for $j \geq 2$ ($y > 0$).
2. The corresponding specifications of J (as $\partial \vec{F} / \partial \vec{x}$; see Fig. A.5).
3. The specific Gaussian elimination scheme used to compute $\vec{\delta}$ (as $-J^{-1} \vec{F}$; see Fig. A.5).

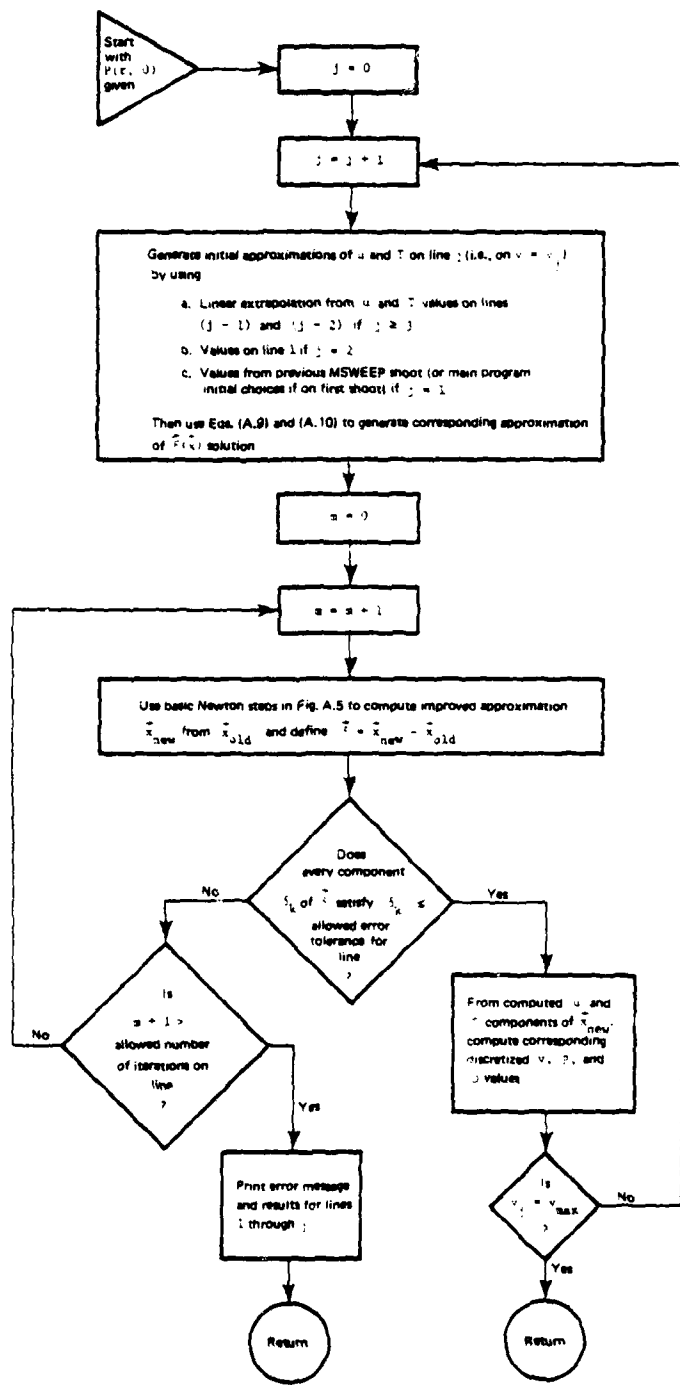


Fig. A.4--Flow chart of subroutine MSWEEP

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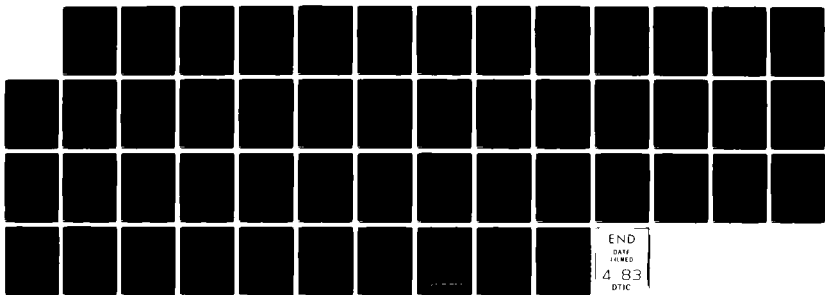
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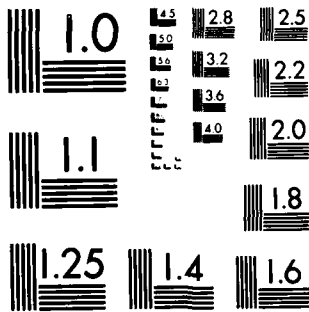
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M-2



MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS 1963 A

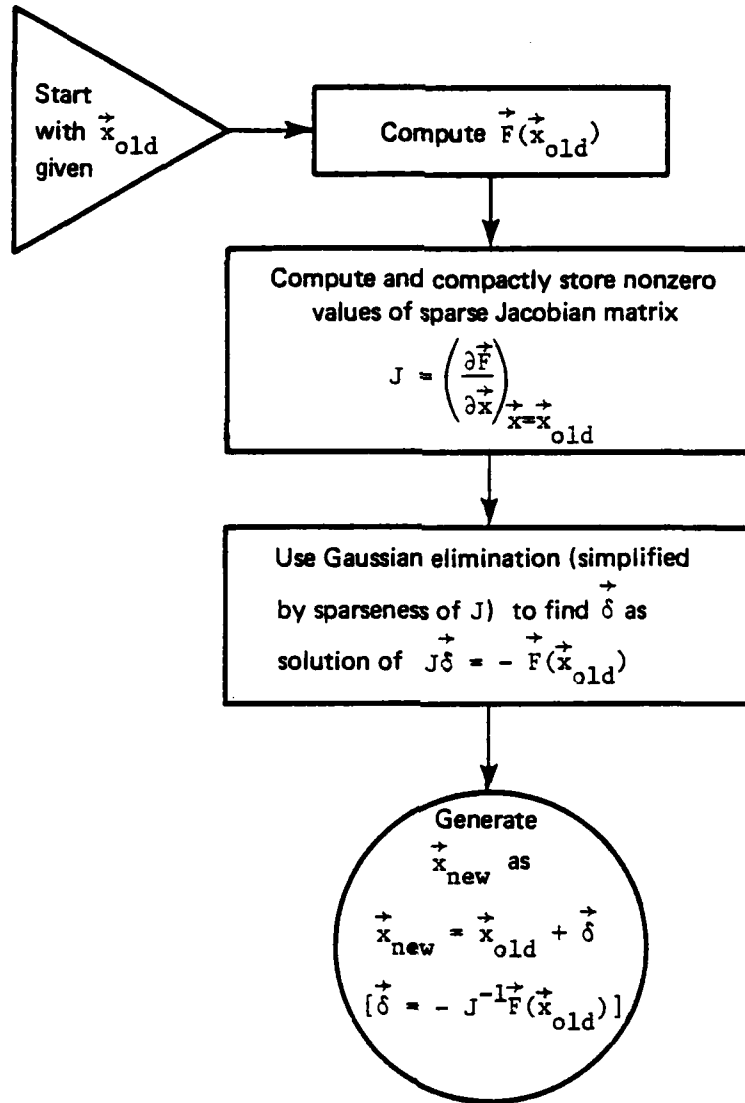


Fig. A.5--Flow chart of Newton steps used in subroutine MSWEEP

For $j = 1$, the usual discretization for second-order ordinary differential equations is used to reduce Eqs. (A.7) to Eq. (A.8). That is, for any dependent variable w , interior derivatives are replaced by the centered difference approximations

$$\begin{aligned} \left(\frac{\partial^2 w}{\partial r^2}\right)_{i,1} &\rightarrow \frac{w_{i+1,1} - 2w_{i,1} + w_{i-1,1}}{(\Delta r)^2}, \\ \left(\frac{\partial w}{\partial r}\right)_{i,1} &\rightarrow \frac{w_{i+1,1} - w_{i-1,1}}{2 \Delta r}. \end{aligned} \quad (\text{A.11})$$

In addition, boundary derivatives are taken as, for example,

$$\left(\frac{\partial w}{\partial r}\right)_{1,1} \rightarrow \frac{w_{2,1} - w_{1,1}}{\Delta r}. \quad (\text{A.12})$$

Thus, as indicated by the corresponding discretization stencil at the bottom of Fig. A.3, no more than three values of u and T are involved in each component equation in Eq. (A.8) [see Eq. (A.9)]--and those values are at successive r_i . For each $j \geq 2$, the Crank-Nicolson scheme to be defined by Eqs. (A.13) and (A.14) is used to reduce Eqs. (A.1), (A.2b), and (A.2d) to Eq. (A.8). Centered differences are also used, with interior and boundary approximation stencils as shown in Fig. A.3, interior approximations being

$$\begin{aligned} \left(\frac{\partial^2 w}{\partial r^2}\right)_{i,j} &\rightarrow \frac{1}{2} \left(\frac{w_{i+1,j} - 2w_{i,j} + w_{i-1,j}}{(\Delta r)^2} + \frac{w_{i+1,j-1} - 2w_{i,j-1} + w_{i-1,j-1}}{(\Delta r)^2} \right), \\ \left(\frac{\partial w}{\partial r}\right)_{i,j} &\rightarrow \frac{1}{2} \left(\frac{w_{i+1,j} - w_{i-1,j}}{2 \Delta r} + \frac{w_{i+1,j-1} - w_{i-1,j-1}}{2 \Delta r} \right), \end{aligned}$$

$$\left(\frac{\partial w}{\partial y}\right)_{i,j} \rightarrow \left(\frac{w_{i,j} - w_{i,j-1}}{\Delta y}\right),$$

$$(w)_{i,j} \rightarrow \frac{1}{2} (w_{i,j} + w_{i,j-1}), \quad (\text{A.13})$$

and the boundary formulas being of the form

$$\left(\frac{\partial w}{\partial r}\right)_{1,j} \rightarrow \frac{1}{2} \left(\frac{w_{2,j} - w_{1,j}}{\Delta r} + \frac{w_{2,j-1} - w_{1,j-1}}{\Delta r}\right),$$

$$(w)_{1,j} \rightarrow \frac{1}{2} (w_{1,j} + w_{1,j-1}). \quad (\text{A.14})$$

As in linear problems [Isaacson and Keller, 1966], the resulting nonlinear scheme has proven to be convergent, stable, and accurate in providing solutions to the turning-region boundary value problem.

In reducing either Eqs. (A.7) or Eqs. (A.1), (A.2b), and (A.2d) to Eq. (A.8) by means of Eqs. (A.11) through (A.14), it proves convenient--both notationally and computationally--to begin by decomposing \vec{x} (for all j) into

$$\vec{u} = \begin{bmatrix} u_{2,j} \\ u_{3,j} \\ \vdots \\ u_{M,j} \\ u_{M+1,j} \end{bmatrix} \quad \text{and} \quad \vec{T} = \begin{bmatrix} T_{2,j} \\ T_{3,j} \\ \vdots \\ T_{M,j} \\ T_{M+1,j} \end{bmatrix}, \quad (\text{A.15})$$

and arranging the corresponding $\vec{F}(\vec{x})$ as

$$\vec{F}_i = \begin{bmatrix} G_2 \\ H_2 \\ G_3 \\ H_3 \\ \vdots \\ \vdots \\ G_M \\ H_M \\ G_{M+1} \\ H_{M+1} \end{bmatrix}, \quad (\text{A.16})$$

where the G_i and H_i represent the components that arise primarily from the discretization of the horizontal momentum and energy equations, respectively, about the grid point (r_i, y_j) . The specific forms of the G_i and H_i are derived shortly. As mentioned above (p. 82), centered differencing results in each G_i and H_i involving at most six of the $2M$ $u_{i,j}$ and $T_{i,j}$. The Jacobian matrix derivatives of

$$\vec{G}(\vec{u}, \vec{T}) = \begin{bmatrix} G_2 \\ G_3 \\ \vdots \\ \vdots \\ G_M \\ G_{M+1} \end{bmatrix} \quad \text{and} \quad \vec{H}(\vec{u}, \vec{T}) = \begin{bmatrix} H_2 \\ H_3 \\ \vdots \\ \vdots \\ H_M \\ H_{M+1} \end{bmatrix} \quad (\text{A.17})$$

with respect to \vec{u} and \vec{T} thus have the following sparse, banded forms:

$$\frac{\partial \vec{G}}{\partial \vec{u}} = \begin{bmatrix} \alpha_{11} & \alpha_{12} & & & & & & & & 0 \\ \alpha_{20} & \alpha_{21} & \alpha_{22} & & & & & & & \\ \alpha_{30} & \alpha_{31} & \alpha_{32} & & & & & & & \\ & & & \alpha_{M-2,0} & \alpha_{M-2,1} & \alpha_{M-2,2} & & & & \\ & & & & \alpha_{M-1,0} & \alpha_{M-1,1} & \alpha_{M-1,2} & & & \\ & & & & & \alpha_{M,0} & \alpha_{M,1} & & & \\ 0 & & & & & & & & & \end{bmatrix},$$

(A.18a)

$$\frac{\partial \vec{G}}{\partial \vec{f}} = \begin{bmatrix} \beta_{11} & \beta_{12} & & & & & & & & 0 \\ \beta_{20} & \beta_{21} & \beta_{22} & & & & & & & \\ \beta_{30} & \beta_{31} & \beta_{32} & & & & & & & \\ & & & \beta_{M-2,0} & \beta_{M-2,1} & \beta_{M-2,2} & & & & \\ & & & & \beta_{M-1,0} & \beta_{M-1,1} & \beta_{M-1,2} & & & \\ & & & & & \beta_{M,0} & \beta_{M,1} & & & \\ 0 & & & & & & & & & \end{bmatrix},$$

(A.18b)

$$\frac{\partial \vec{H}}{\partial \vec{u}} = \begin{bmatrix} \gamma_{11} & \gamma_{12} & & & & & 0 \\ \gamma_{20} & \gamma_{21} & \gamma_{22} & & & & \\ & \gamma_{30} & \gamma_{31} & \gamma_{32} & & & \\ & & & & \gamma_{M-2,0} & \gamma_{M-2,1} & \gamma_{M-2,2} \\ 0 & & & & & \gamma_{M-1,0} & \gamma_{M-1,1} & \gamma_{M-1,2} \\ & & & & & & \gamma_{M,0} & \gamma_{M,1} \end{bmatrix},$$

(A.18c)

$$\frac{\partial \vec{H}}{\partial \vec{T}} = \begin{bmatrix} \delta_{11} & \delta_{12} & & & & & 0 \\ \delta_{20} & \delta_{21} & \delta_{22} & & & & \\ & \delta_{30} & \delta_{31} & \delta_{32} & & & \\ & & & & \delta_{M-2,0} & \delta_{M-2,1} & \delta_{M-2,2} \\ 0 & & & & & \delta_{M-1,0} & \delta_{M-1,1} & \delta_{M-1,2} \\ & & & & & & \delta_{M,0} & \delta_{M,1} \end{bmatrix}.$$

(A.18d)

From Eqs. (A.10) and (A.15) through (A.18), the $J = \partial \vec{F} / \partial \vec{x}$ Jacobian is then easily computed from the nonzero elements in the Jacobians of Eqs. (A.13) as

x_{11}	δ_{11}	γ_{12}	δ_{12}															
x_{12}	γ_{11}	γ_{12}	δ_{12}	1														
x_{21}	δ_{20}	x_{21}	δ_{21}	γ_{22}	δ_{22}													
x_{22}	γ_{20}	γ_{21}	δ_{21}	γ_{22}	δ_{22}	0												
	x_{30}	δ_{30}	γ_{31}	δ_{31}	x_{32}	δ_{32}												
	γ_{30}	δ_{30}	γ_{31}	δ_{31}	γ_{32}	δ_{32}	1											
								$\alpha_{M-2,0}$	$\beta_{M-2,0}$	$\gamma_{M-2,1}$	$\delta_{M-2,1}$	$\alpha_{M-2,2}$	$\beta_{M-2,2}$					
								$\gamma_{M-2,0}$	$\delta_{M-2,0}$	$\gamma_{M-2,1}$	$\delta_{M-2,1}$	$\gamma_{M-2,2}$	$\delta_{M-2,2}$	0				
									0	$\alpha_{M-1,0}$	$\beta_{M-1,0}$	$\gamma_{M-1,1}$	$\delta_{M-1,1}$	$\alpha_{M-1,2}$	$\beta_{M-1,2}$	$\gamma_{M-1,2}$	$\delta_{M-1,2}$	
										$\gamma_{M-1,0}$	$\delta_{M-1,0}$	$\gamma_{M-1,1}$	$\delta_{M-1,1}$	$\gamma_{M-1,2}$	$\delta_{M-1,2}$			
											0	$\alpha_{M,0}$	$\beta_{M,0}$	$\alpha_{M,1}$	$\beta_{M,1}$	$\alpha_{M,2}$	$\beta_{M,2}$	
												$\alpha_{M,0}$	$\beta_{M,0}$	$\alpha_{M,1}$	$\beta_{M,1}$	$\alpha_{M,2}$	$\beta_{M,2}$	

(A.19)

DIFFERENCE EQUATIONS AND JACOBIAN ELEMENTS

Specific forms for \vec{G} and \vec{H} , and hence for \vec{F} and the various α , β , γ , and δ coefficients in Eqs. (A.18) and (A.19), are derived as follows. Subject to Eqs. (A.11), the first two lines of Eqs. (A.7) are approximated at $r = r_1$, $2 \leq i \leq M$, by

$$\begin{aligned}
 \frac{u_{i,1}}{T_{i,1}} \left(\frac{u_{i+1,1} - u_{i-1,1}}{2 \Delta r} \right) &= - \left(\frac{P_{i+1,1} - P_{i-1,1}}{2 \Delta r} \right) \\
 &+ M_1 \left(\frac{u_{i+1,1} - 2u_{i,1} + u_{i-1,1}}{(\Delta r)^2} \right) \\
 &+ \frac{1}{r_i} \left(\frac{u_{i+1,1} - u_{i-1,1}}{2 \Delta r} \right) - \frac{u_{i,1}}{r_i^2} \Bigg) , \\
 \frac{u_{i,1}}{T_{i,1}} \left(\frac{T_{i+1,1} - T_{i-1,1}}{2 \Delta r} \right) &= q(r_i, 0) - \sigma(T_{i,1}^4 - 1) \\
 &+ K_1 \left(\frac{T_{i+1,1} - 2T_{i,1} + T_{i-1,1}}{(\Delta r)^2} \right) \\
 &+ \frac{1}{r_i} \left(\frac{T_{i+1,1} - T_{i-1,1}}{2 \Delta r} \right) \Bigg) . \tag{A.20}
 \end{aligned}$$

From Eqs. (A.2b), (A.2d), and (A.12), we then also have

$$u_{1,1} = 0 , \quad T_{1,1} = T_{2,1} , \tag{A.21}$$

and

$$\begin{aligned}
 M_1 \left(\frac{u_{M+1,1} - u_{M,1}}{\Delta r} \right) &= \left(\frac{u_{M+1,1}}{T_{M+1,1}} \right)^2 (T_{M+1,1} - 1) + P_{M+1,1} , \\
 K_1 \left(\frac{T_{M+1,1} - T_{M,1}}{\Delta r} \right) &= \frac{u_{M+1,1}}{T_{M+1,1}} (T_{M+1,1} - 1) . \tag{A.22}
 \end{aligned}$$

For $j = 1$, we thus have

$$\begin{aligned}
 G_2 &= \frac{M_1}{\Delta r} (u_{3,1} - 2u_{2,1}) - \frac{1}{2} \left(\frac{u_{2,1}}{T_{2,1}} - \frac{M_1}{r_2} \right) u_{3,1} - \frac{1}{2} (P_{3,1} - P_{1,1}) \\
 &\quad - \left(\frac{M_1}{r_2} \right) u_{2,1} \Delta r, \\
 H_2 &= \frac{K_1}{\Delta r} (T_{3,1} - T_{2,1}) - \frac{1}{2} \left(\frac{u_{2,1}}{T_{2,1}} - \frac{K_1}{r_2} \right) (T_{3,1} - T_{2,1}) \\
 &\quad + \left(q(r_2, 0) - \sigma(T_{2,1}^4 - 1) \right) \Delta r; \tag{A.23}
 \end{aligned}$$

$$\begin{aligned}
 G_i &= \frac{M_1}{\Delta r} (u_{i+1,1} - 2u_{i,1} + u_{i-1,1}) - \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} - \frac{M_1}{r_i} \right) (u_{i+1,1} - u_{i-1,1}) \\
 &\quad - \frac{1}{2} (P_{i+1,1} - P_{i-1,1}) - M_1 \left(\frac{u_{i,1}}{r_i} \right) \frac{\Delta r}{r_i}, \\
 H_i &= \frac{K_1}{\Delta r} (T_{i+1,1} - 2T_{i,1} + T_{i-1,1}) - \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} - \frac{K_1}{r_i} \right) (T_{i+1,1} - T_{i-1,1}) \\
 &\quad + \left(q(r_i, 0) - \sigma(T_{i,1}^4 - 1) \right) \Delta r \tag{A.24}
 \end{aligned}$$

for $3 \leq i \leq M$; and

$$G_{M+1} = \frac{M_1}{\Delta r} (u_{M+1,1} - u_{M,1}) - \left(\frac{u_{M+1,1}}{T_{M+1,1}} \right)^2 (T_{M+1,1} - 1) - P_{M+1,1},$$

$$H_{M+1} = \frac{K_1}{\Delta r} (T_{M+1,1} - T_{M,1}) - \frac{u_{M+1,1}}{T_{M+1,1}} (T_{M+1,1} - 1) . \quad (\text{A.25})$$

From Eqs. (A.23) through (A.25), the Jacobian elements in Eqs. (A.18) and (A.19) are then found to be

$$\alpha_{11} = -3 \left(\frac{M_1}{\Delta r} \right) - \frac{1}{2} \left(\frac{u_{3,1}}{T_{2,1}} \right) ,$$

$$\alpha_{12} = \frac{3}{2} \left(\frac{M_1}{\Delta r} \right) - \frac{1}{2} \left(\frac{u_{2,1}}{T_{2,1}} \right) ,$$

$$\beta_{11} = \frac{1}{2} \left(\frac{u_{2,1} u_{3,1}}{T_{2,1}^2} \right) ,$$

$$\beta_{12} = 0 ,$$

$$\gamma_{11} = -\frac{1}{2} \left(\frac{T_{3,1}}{T_{2,1}} - 1 \right) ,$$

$$\gamma_{12} = 0 ,$$

$$\delta_{11} = -\frac{3}{2} \left(\frac{K_1}{\Delta r} \right) + \frac{1}{2} \left(\frac{u_{2,1} T_{3,1}}{T_{2,1}^2} \right) - 4\sigma T_{2,1}^3 (\Delta r) ,$$

$$\delta_{12} = \frac{3}{2} \left(\frac{K_1}{\Delta r} \right) - \frac{1}{2} \left(\frac{u_{2,1}}{T_{2,1}} \right) ; \quad (\text{A.26})$$

$$\alpha_{i-1,0} = \frac{M_1}{\Delta r} \left(1 - \frac{\Delta r}{2r_i} \right) + \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} \right) ,$$

$$\alpha_{i-1,1} = - \frac{M_1}{\Delta r} \left(2 + \left(\frac{\Delta r}{r_i} \right)^2 \right) - \frac{1}{2} \left(\frac{u_{i+1,1} - u_{i-1,1}}{T_{i,1}} \right) ,$$

$$\alpha_{i-1,2} = \frac{M_1}{\Delta r} \left(1 + \frac{\Delta r}{2r_i} \right) - \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} \right) ,$$

$$\beta_{i-1,0} = 0 ,$$

$$\beta_{i-1,1} = \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}^2} \right) (u_{i+1,1} - u_{i-1,1}) ,$$

$$\beta_{i-1,2} = 0 ,$$

$$\gamma_{i-1,0} = 0 ,$$

$$\gamma_{i-1,1} = - \frac{1}{2} \left(\frac{T_{i+1,1} - T_{i-1,1}}{T_{i,1}} \right) ,$$

$$\gamma_{i-1,2} = 0 ,$$

$$\delta_{i-1,0} = \frac{K_1}{\Delta r} \left(1 - \frac{\Delta r}{2r_i} \right) + \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} \right) ,$$

$$\delta_{i-1,1} = - 2 \left(\frac{K_1}{\Delta r} \right) + \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}^2} \right) (T_{i+1,1} - T_{i-1,1}) - 4\sigma T_{i,1}^3 (\Delta r) ,$$

$$\delta_{i-1,2} = \frac{K_1}{\Delta r} \left(1 + \frac{\Delta r}{2r_i} \right) - \frac{1}{2} \left(\frac{u_{i,1}}{T_{i,1}} \right)$$

(A.27)

for $3 \leq i \leq M$; and

$$\alpha_{M,0} = -\frac{M_1}{\Delta r},$$

$$\alpha_{M,1} = \frac{M_1}{\Delta r} - 2 \left(\frac{u_{M+1,1}}{T_{M+1,1}} \right) \left(1 - \frac{1}{T_{M+1,1}} \right),$$

$$\beta_{M,0} = 0,$$

$$\beta_{M,1} = \left(\frac{u_{M+1,1}}{T_{M+1,1}} \right)^2 \left(1 - \frac{2}{T_{M+1,1}} \right),$$

$$\gamma_{M,0} = -1 + \frac{1}{T_{M+1,1}},$$

$$\gamma_{M,1} = 0,$$

$$\delta_{M,0} = -\frac{K_1}{\Delta r},$$

$$\delta_{M,1} = \frac{K_1}{\Delta r} - \frac{u_{M+1,1}}{T_{M+1,1}^2}. \quad (\text{A.28})$$

For $j \geq 2$, the derivation of relevant forms for \vec{G} and \vec{H} begins with a preliminary rewriting of Eqs. (A.1) in the following equivalent form:

$$\frac{1}{r} \frac{\partial}{\partial r} (r \rho u) + \frac{\partial}{\partial y} (\rho v) = 0,$$

$$\frac{1}{r} \frac{\partial}{\partial r} (r \rho u^2) + \frac{\partial P}{\partial r} + \frac{\partial}{\partial y} (\rho u v) = M_1 \left(\frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \frac{u}{r^2} \right),$$

$$\frac{\partial P}{\partial y} + \Delta p = 0 ,$$

$$\frac{1}{r} \frac{\partial}{\partial r} (ru) + \frac{\partial v}{\partial y} \equiv \frac{1}{r} \frac{\partial}{\partial r} (r\rho uT) + \frac{\partial}{\partial y} (\rho vT) = K_1 \left(\frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} \right) + q(r, y) - \sigma(T^4 - 1) ,$$

$$\rho T = 1 .$$

(A.29)

Subject to Eqs. (A.13), this set is approximated at each interior grid point (r_i, y_j) , $2 \leq i \leq M$, of line j , $2 \leq j \leq N + 1$ by the finite difference system

$$\frac{1}{4 \Delta r} \left\{ \frac{r_{i+1}}{r_i} \left(\left(\frac{u}{T} \right)_{i+1,j} + \left(\frac{u}{T} \right)_{i+1,j-1} \right) - \frac{r_{i-1}}{r_i} \left(\left(\frac{u}{T} \right)_{i-1,j} + \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\} + \frac{1}{\Delta y} \left(\left(\frac{v}{T} \right)_{i,j} - \left(\frac{v}{T} \right)_{i,j-1} \right) = 0 , \quad (\text{A.30a})$$

$$\begin{aligned} & \frac{1}{4 \Delta r} \left\{ \frac{r_{i+1}}{r_i} \left(\left(\frac{u^2}{T} \right)_{i+1,j} + \left(\frac{u^2}{T} \right)_{i+1,j-1} \right) - \frac{r_{i-1}}{r_i} \left(\left(\frac{u^2}{T} \right)_{i-1,j} + \left(\frac{u^2}{T} \right)_{i-1,j-1} \right) \right\} \\ & + \frac{1}{4 \Delta r} (P_{i+1,j} + P_{i+1,j-1} - P_{i-1,j} - P_{i-1,j-1}) \\ & + \frac{1}{\Delta y} \left(\left(\frac{uv}{T} \right)_{i,j} - \left(\frac{uv}{T} \right)_{i,j-1} \right) \\ & = M_1 \left(\frac{u_{i+1,j} - 2u_{i,j} + u_{i-1,j} + u_{i+1,j-1} - 2u_{i,j-1} + u_{i-1,j-1}}{2(\Delta r)^2} \right. \\ & + \frac{1}{r_i} \left(\frac{u_{i+1,j} - u_{i-1,j} + u_{i+1,j-1} - u_{i-1,j-1}}{4 \Delta r} \right) \\ & \left. - \frac{u_{i,j} + u_{i,j-1}}{2r_i^2} \right) , \quad (\text{A.30b}) \end{aligned}$$

$$\frac{1}{\Delta y} (P_{i,j} - P_{i,j-1}) = -\frac{A}{2} \left(\left(\frac{1}{T} \right)_{i,j} + \left(\frac{1}{T} \right)_{i,j-1} \right), \quad (\text{A.30c})$$

$$\begin{aligned} & \frac{1}{4 \Delta r} \left(\frac{r_{i+1}}{r_i} (u_{i+1,j} + u_{i+1,j-1}) - \frac{r_{i-1}}{r_i} (u_{i-1,j} + u_{i-1,j-1}) \right) \\ & + \frac{1}{\Delta y} (v_{i,j} - v_{i,j-1}) \\ & = K_1 \left(\frac{T_{i+1,j} - 2T_{i,j} + T_{i-1,j} + T_{i+1,j-1} - 2T_{i,j-1} + T_{i-1,j-1}}{2(\Delta r)^2} \right. \\ & \quad \left. + \frac{1}{r_i} \left(\frac{T_{i+1,j} - T_{i-1,j} + T_{i+1,j-1} - T_{i-1,j-1}}{4 \Delta r} \right) \right) \\ & \quad + \frac{1}{2} (q(r_i, y_j) + q(r_i, y_{j-1})) + \sigma \left(\left(\frac{T_{i,j} + T_{i,j-1}}{2} \right)^4 - 1 \right), \end{aligned} \quad (\text{A.30d})$$

ρ having been eliminated for convenience. This system is simplified by solving Eqs. (A.30a) and (A.30c) for $v_{i,j}$ and $P_{i,j}$ in terms of the various u and T on line j and variables on line $j - 1$ as

$$\begin{aligned} v_{i,j} &= \left(\frac{v}{T} \right)_{i,j-1} T_{i,j} - \frac{1}{4} \left(\frac{\Delta y}{\Delta r} \right) \left\{ \frac{r_{i+1}}{r_i} \left(\left(\frac{u}{T} \right)_{i+1,j} + \left(\frac{u}{T} \right)_{i+1,j-1} \right) \right. \\ & \quad \left. - \frac{r_{i-1}}{r_i} \left(\left(\frac{u}{T} \right)_{i-1,j} + \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\} T_{i,j}, \\ P_{i,j} &= P_{i,j-1} - \frac{A}{2} \left(\left(\frac{1}{T} \right)_{i,j} + \left(\frac{1}{T} \right)_{i,j-1} \right) \Delta y, \end{aligned} \quad (\text{A.31})$$

and substituting those results into Eqs. (A.30b) and (A.30d) to generate (for $2 \leq i \leq M$)

$$\begin{aligned}
 G_i \equiv & \frac{M_1}{2 \Delta r} (u_{i+1,j} - 2u_{i,j} + u_{i-1,j}) - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \left(\frac{u^2}{T} \right)_{i+1,j} - \frac{r_{i-1}}{r_i} \left(\frac{u^2}{T} \right)_{i-1,j} \right. \\
 & \left. - \frac{M_1}{r_i} (u_{i+1,j} - u_{i-1,j}) + 2 \left(\frac{M_1}{r_i^2} \right) u_{i,j} (\Delta r) - A \frac{\Delta y}{2} \left(\frac{1}{T_{i+1,j}} - \frac{1}{T_{i-1,j}} \right) \right) \\
 & - \left(\frac{\Delta r}{\Delta y} \right) u_{i,j} \left\{ \left(\frac{v}{T} \right)_{i,j-1} - \frac{\Delta y}{4 \Delta r} \left(\frac{r_{i+1}}{r_i} \left(\frac{u}{T} \right)_{i+1,j} - \frac{r_{i-1}}{r_i} \left(\frac{u}{T} \right)_{i-1,j} \right. \right. \\
 & \left. \left. + \frac{r_{i+1}}{r_i} \left(\frac{u}{T} \right)_{i+1,j-1} - \frac{r_{i-1}}{r_i} \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\} + R_i = 0 ,
 \end{aligned}$$

$$\begin{aligned}
 H_i \equiv & \frac{K_1}{2 \Delta r} (T_{i+1,j} - 2T_{i,j} + T_{i-1,j}) - \frac{1}{4} \left(\left(\frac{r_{i+1}}{r_i} \right) u_{i+1,j} - \left(\frac{r_{i-1}}{r_i} \right) u_{i-1,j} \right. \\
 & \left. - \frac{K_1}{r_i} (T_{i+1,j} - T_{i-1,j}) \right) - \left(\frac{\Delta r}{\Delta y} \right) T_{i,j} \left\{ \left(\frac{v}{T} \right)_{i,j-1} - \frac{\Delta y}{4 \Delta r} \right. \\
 & \left. \times \left(\frac{r_{i+1}}{r_i} \left(\frac{u}{T} \right)_{i+1,j} - \frac{r_{i-1}}{r_i} \left(\frac{u}{T} \right)_{i-1,j} + \frac{r_{i+1}}{r_i} \left(\frac{u}{T} \right)_{i+1,j-1} - \frac{r_{i-1}}{r_i} \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\} \\
 & + \sigma \left(\left(\frac{T_{i,j} + T_{i,j-1}}{2} \right)^4 - 1 \right) \Delta r + \Pi_i = 0 , \tag{A.32}
 \end{aligned}$$

where

$$\begin{aligned}
 R_i = & \frac{M_1}{2 \Delta r} (u_{i+1,j-1} - 2u_{i,j-1} + u_{i-1,j-1}) - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \left(\frac{u^2}{T} \right)_{i+1,j-1} \right. \\
 & - \frac{r_{i-1}}{r_i} \left(\frac{u^2}{T} \right)_{i-1,j-1} - \frac{M_1}{r_i} (u_{i+1,j-1} - u_{i-1,j-1}) + 2 \left(\frac{M_1}{r_i^2} \right) u_{i,j-1} \Delta r \\
 & + 2(P_{i+1,j-1} - P_{i-1,j-1}) - A \frac{\Delta y}{2} \left(\frac{1}{T_{i+1,j-1}} - \frac{1}{T_{i-1,j-1}} \right) \\
 & + \frac{\Delta r}{\Delta y} \left(\frac{uv}{T} \right)_{i,j-1}, \\
 \Pi_i = & \frac{K_1}{2 \Delta r} (T_{i+1,j-1} - 2T_{i,j-1} + T_{i-1,j-1}) - \frac{1}{4} \left(\left(\frac{r_{i+1}}{r_i} \right) u_{i+1,j-1} \right. \\
 & - \left. \left(\frac{r_{i-1}}{r_i} \right) u_{i-1,j-1} - \frac{K_1}{r_i} (T_{i+1,j-1} - T_{i-1,j-1}) \right) + \frac{\Delta r}{\Delta y} v_{i,j-1} \\
 & + \frac{1}{2} (q(r_i, y_j) + q(r_i, y_{j-1})) \Delta r. \tag{A.33}
 \end{aligned}$$

As outlined in Fig. A.4, a recursive Crank-Nicolson scheme is employed in subroutine MSWEEP to generate a numerical solution to Eqs. (A.1), (A.2a), (A.2b), and (A.2d), subject to a given $P(r, 0)$. For $j = 2, 3, \dots, N + 1$, the solution on line j is found using solution data on line $j - 1$, the solution for $j = 1$ being used to start the recursion. For each $j \geq 2$, the relevant forms of G_i and H_i (as components of \vec{F}), $2 \leq i \leq M$, are therefore exactly those in Eqs. (A.32) [cf. Eqs. (A.24)], variables with $j - 1$ suffices being known. From Eqs. (A.2b), (A.2d), and (A.14), we also have

$$u_{1,j} = 0, \quad T_{1,j} = T_{2,j}, \quad (\text{A.34})$$

and

$$\begin{aligned} M_1 \left(\frac{u_{M+1,j} - u_{M,j} + u_{M+1,j-1} - u_{M,j-1}}{2 \Delta r} \right) &= \left(\frac{u_{M+1,j} + u_{M+1,j-1}}{T_{M+1,j} + T_{M+1,j-1}} \right)^2 \\ &\times \left(\frac{T_{M+1,j} + T_{M+1,j-1} - 2}{2} \right) \\ &+ \left(\frac{P_{M+1,j} + P_{M+1,j-1}}{2} \right) \\ &+ \frac{A}{2} (y_j + y_{j-1}) \quad , \\ K_1 \left(\frac{T_{M+1,j} - T_{M,j} + T_{M+1,j-1} - T_{M,j-1}}{2 \Delta r} \right) &= \left(\frac{u_{M+1,j} + u_{M+1,j-1}}{T_{M+1,j} + T_{M+1,j-1}} \right) \\ &\times \left(\frac{T_{M+1,j} + T_{M+1,j-1} - 2}{2} \right) . \end{aligned}$$

(A.35)

Therefore, for $j \geq 2$,

$$\begin{aligned} G_{M+1} &= \frac{M_1}{2 \Delta r} (u_{M+1,j} - u_{M,j} + u_{M+1,j-1} - u_{M,j-1}) - E_j^2 \left(\frac{T_{M+1,j} + T_{M+1,j-1} - 2}{2} \right) \\ &- P_{M+1,j-1} + \frac{A}{4} \left(\frac{1}{T_{M+1,j}} + \frac{1}{T_{M+1,j-1}} \right) \Delta y - \frac{A}{2} (y_j + y_{j-1}) \quad , \\ H_{M+1} &= \frac{K_1}{2 \Delta r} (T_{M+1,j} - T_{M,j} + T_{M+1,j-1} - T_{M,j-1}) \\ &- E_j \left(\frac{T_{M+1,j} + T_{M+1,j-1} - 2}{2} \right) \quad , \end{aligned} \quad (\text{A.36})$$

where

$$E_j = \frac{u_{M+1,j} + u_{M+1,j-1}}{T_{M+1,j} + T_{M+1,j-1}} . \quad (\text{A.37})$$

From Eqs. (A.32), (A.34), and (A.36), the Jacobian elements in Eqs. (A.18) and (A.19) are then as follows for each $j \geq 2$:

$$\alpha_{11} = -\frac{3}{2} \left(\frac{M_1}{\Delta r} \right) - \left\{ \frac{\Delta r}{\Delta y} \left(\frac{v}{T} \right)_{2,j-1} - \frac{2}{4} \left(\left(\frac{u}{T} \right)_{3,j} + \left(\frac{u}{T} \right)_{3,j-1} \right) \right\} ,$$

$$\alpha_{12} = \frac{3}{4} \left(\frac{M_1}{\Delta r} \right) - \frac{2}{4} \left(2 \left(\frac{u}{T} \right)_{3,j} - \frac{u_{2,j}}{T_{3,j}} \right) ,$$

$$\beta_{11} = \frac{1}{4} \left(A \left(\frac{\Delta y}{2} \right) \left(\frac{1}{T_{2,j}} \right)^2 \right) ,$$

$$\beta_{12} = \frac{1}{4} \left(2 \left(\frac{u}{T} \right)_{3,j}^2 - A \left(\frac{\Delta y}{2} \right) \left(\frac{1}{T_{3,j}} \right)^2 - 2u_{2,j} \left(\frac{u}{T^2} \right)_{3,j} \right) ,$$

$$\gamma_{11} = 0 ,$$

$$\gamma_{12} = -\frac{2}{4} \left(1 - \frac{T_{2,j}}{T_{3,j}} \right) ,$$

$$\delta_{11} = -\frac{3}{4} \left(\frac{K_1}{\Delta r} \right) - \left\{ \frac{\Delta r}{\Delta y} \left(\frac{v}{T} \right)_{2,j-1} - \frac{2}{4} \left(\left(\frac{u}{T} \right)_{3,j} + \left(\frac{u}{T} \right)_{3,j-1} \right) \right\} \\ + 2\sigma \left(\frac{T_{2,j} + T_{2,j-1}}{2} \right)^3 \Delta r ,$$

$$\delta_{12} = \frac{3}{4} \left(\frac{K_1}{\Delta r} \right) - \frac{2}{4} \left(T_{2,j} \left(\frac{u}{T^2} \right)_{3,j} \right) ; \quad (\text{A.38})$$

$$\alpha_{i-1,0} = \frac{M_1}{2 \Delta r} \left(1 - \frac{\Delta r}{2r_i} \right) + \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \right) \left(2 \left(\frac{u}{T} \right)_{i-1,j} - \frac{u_{i,j}}{T_{i-1,j}} \right),$$

$$\begin{aligned} \alpha_{i-1,1} = & - \frac{M_1}{2 \Delta r} \left(2 + \left(\frac{\Delta r}{r_i} \right)^2 \right) - \left(\frac{\Delta r}{\Delta y} \left(\frac{v}{T} \right)_{i,j-1} \right. \\ & - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \right) \left(\left(\frac{u}{T} \right)_{i+1,j} + \left(\frac{u}{T} \right)_{i+1,j-1} \right) \\ & \left. + \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \right) \left(\left(\frac{u}{T} \right)_{i-1,j} + \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\}, \end{aligned}$$

$$\alpha_{i-1,2} = \frac{M_1}{2 \Delta r} \left(1 + \frac{\Delta r}{2r_i} \right) - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \right) \left(2 \left(\frac{u}{T} \right)_{i+1,j} - \frac{u_{i,j}}{T_{i+1,j}} \right),$$

$$\begin{aligned} \beta_{i-1,0} = & - \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \left(\frac{u}{T} \right)_{i-1,j}^2 - A \left(\frac{\Delta y}{2} \right) \left(\frac{1}{T_{i-1,j}} \right)^2 \right. \\ & \left. - \frac{r_{i-1}}{r_i} u_{i,j} \left(\frac{u}{T} \right)_{i-1,j} \right), \end{aligned}$$

$$\beta_{i-1,1} = 0,$$

$$\begin{aligned} \beta_{i-1,2} = & \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \left(\frac{u}{T} \right)_{i+1,j}^2 - A \left(\frac{\Delta y}{2} \right) \left(\frac{1}{T_{i+1,j}} \right)^2 \right. \\ & \left. - \frac{r_{i+1}}{r_i} u_{i,j} \left(\frac{u}{T} \right)_{i+1,j} \right), \end{aligned}$$

$$\gamma_{i-1,0} = \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \right) \left(1 - \frac{T_{i,j}}{T_{i-1,j}} \right),$$

$$\gamma_{i-1,1} = 0,$$

$$\gamma_{i-1,2} = -\frac{1}{4} \left(\frac{r_{i+1}}{r_i} \right) \left(1 - \frac{T_{i,j}}{T_{i+1,j}} \right),$$

$$\delta_{i-1,0} = \frac{K_1}{2 \Delta r} \left(1 - \frac{\Delta r}{2r_i} \right) + \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \right) \left(T_{i,j} \left(\frac{u}{T^2} \right)_{i-1,j} \right),$$

$$\begin{aligned} \delta_{i-1,1} = & -2 \left(\frac{K_1}{2 \Delta r} \right) - \left\{ \frac{\Delta r}{\Delta y} \left(\frac{v}{T} \right)_{i,j-1} - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \right) \left(\left(\frac{u}{T} \right)_{i+1,j} + \left(\frac{u}{T} \right)_{i+1,j-1} \right) \right. \\ & \left. + \frac{1}{4} \left(\frac{r_{i-1}}{r_i} \right) \left(\left(\frac{u}{T} \right)_{i-1,j} + \left(\frac{u}{T} \right)_{i-1,j-1} \right) \right\} + 2\sigma \left(\frac{T_{i,j} + T_{i,j-1}}{2} \right)^3 \Delta r, \end{aligned}$$

$$\delta_{i-1,2} = \frac{K_1}{2 \Delta r} \left(1 + \frac{\Delta r}{2r_i} \right) - \frac{1}{4} \left(\frac{r_{i+1}}{r_i} \right) \left(T_{i,j} \left(\frac{u}{T^2} \right)_{i+1,j} \right) \quad (\text{A.39})$$

for $3 \leq i \leq M$; and

$$\alpha_{M,0} = -\frac{M_1}{2 \Delta r},$$

$$\alpha_{M,1} = \frac{M_1}{2 \Delta r} - 2E_j \left(\frac{1}{2} - \frac{1}{T_{M+1,j} + T_{M+1,j-1}} \right),$$

$$\beta_{M,0} = 0,$$

$$\beta_{M,1} = E_j^2 \left(\frac{1}{2} - \frac{2}{T_{M+1,j} + T_{M+1,j-1}} \right) - \frac{A}{4} \left(\frac{1}{T_{M+1,j}^2} \right) \Delta y.$$

$$Y_{M,0} = 0 ,$$

$$Y_{M,1} = -\frac{1}{2} + \frac{1}{T_{M+1,j} + T_{M+1,j-1}} ,$$

$$\delta_{M,0} = -\frac{K_1}{2 \Delta r} ,$$

$$\delta_{M,1} = \frac{K_1}{2 \Delta r} - \frac{E_j}{T_{M+1,j} + T_{M+1,j-1}} . \quad (A.40)$$

The detailed specification of \vec{G} , \vec{H} , and $J (= \partial \vec{F} / \partial \vec{x})$ is now complete for all cases. The equation $J\vec{\delta} = -\vec{F}$ is solved for $\vec{\delta}$ using Gaussian elimination [Isaacson and Keller, 1966]. For efficiency, standard sparse-matrix methods are employed: only nonzero elements of J [see Eq. (A.19)] are stored, and operations involving zero elements are automatically omitted.

PROGRAM LISTING

FORTRAN listings of all parts of the computer code are now presented. Table A.2 lists the main program, and Tables A.3, A.4, and A.5 list the subroutines BCFUNC, SPRINT, and MSWEEP, respectively.

Table A.2
LISTING OF MAIN PROGRAM

ADDRESS	OPERATION	COMMENT	LINE NO.
000001	START	START OF PROGRAM	1
000002	LOAD	LOAD DATA	2
000003	STORE	STORE DATA	3
000004	ADD	ADD DATA	4
000005	SUBTRACT	SUBTRACT DATA	5
000006	MULTIPLY	MULTIPLY DATA	6
000007	DIVIDE	DIVIDE DATA	7
000008	IF	IF STATEMENT	8
000009	GO TO	GO TO STATEMENT	9
000010	CALL	CALL SUBROUTINE	10
000011	RETURN	RETURN FROM SUBROUTINE	11
000012	STOP	STOP PROGRAM	12
000013	END	END OF PROGRAM	13
000014			14
000015			15
000016			16
000017			17
000018			18
000019			19
000020			20
000021			21
000022			22
000023			23
000024			24
000025			25
000026			26
000027			27
000028			28
000029			29
000030			30
000031			31
000032			32
000033			33
000034			34
000035			35
000036			36
000037			37
000038			38
000039			39
000040			40
000041			41
000042			42
000043			43
000044			44
000045			45
000046			46
000047			47
000048			48
000049			49
000050			50
000051			51
000052			52
000053			53
000054			54
000055			55
000056			56
000057			57

Table A.2--Continued

FORMAL VALUE	ALPHABETIC SYMBOL	SYMBOL	UNIT	DESCRIPTION	NO.
58	000540F			000540F	58
59	000541F			000541F	59
60	000542F			000542F	60
61	000543F			000543F	61
62	000544F			000544F	62
63	000545F			000545F	63
64	000546F			000546F	64
65	000547F			000547F	65
66	000548F			000548F	66
67	000549F			000549F	67
68	000550F			000550F	68
69	000551F			000551F	69
70	000552F			000552F	70
71	000553F			000553F	71
72	000554F			000554F	72
73	000555F			000555F	73
74	000556F			000556F	74
75	000557F			000557F	75
76	000558F			000558F	76
77	000559F			000559F	77
78	000560F			000560F	78
79	000561F			000561F	79
80	000562F			000562F	80
81	000563F			000563F	81
82	000564F			000564F	82
83	000565F			000565F	83
84	000566F			000566F	84
85	000567F			000567F	85
86	000568F			000568F	86
87	000569F			000569F	87
88	000570F			000570F	88
89	000571F			000571F	89
90	000572F			000572F	90
91	000573F			000573F	91
92	000574F			000574F	92
93	000575F			000575F	93
94	000576F			000576F	94
95	000577F			000577F	95
96	000578F			000578F	96
97	000579F			000579F	97
98	000580F			000580F	98
99	000581F			000581F	99
100	000582F			000582F	100
101	000583F			000583F	101
102	000584F			000584F	102
103	000585F			000585F	103
104	000586F			000586F	104
105	000587F			000587F	105
106	000588F			000588F	106
107	000589F			000589F	107
108	000590F			000590F	108
109	000591F			000591F	109
110	000592F			000592F	110
111	000593F			000593F	111
112	000594F			000594F	112
113	000595F			000595F	113
114	000596F			000596F	114

Table A.2--Continued

FORTRAN-77 ID	FORTRAN-77 ID	FORTRAN-77 ID	FORTRAN-77 ID	FORTRAN-77 ID
115	000680E	CALL SPZ(1000)		115
116				116
117	000681E	IF (P(1) .EQ. 1) GO TO 900		117
118				118
119				119
120				120
121				121
122	000682E	WRITE(44)A(1)		122
123	000683E	GO TO 110		123
124	000684E	WRITE(44)A(1)		124
125	000685E	WRITE(44)A(1)		125
126	000686E	WRITE(44)A(1)		126
127	000687E	WRITE(44)A(1)		127
128	000688E	WRITE(44)A(1)		128
129				129
130	000689E	WRITE(44)A(1)		130
131	000690E	WRITE(44)A(1)		131
132	000691E	WRITE(44)A(1)		132
133	000692E	WRITE(44)A(1)		133
134	000693E	WRITE(44)A(1)		134
135	000694E	WRITE(44)A(1)		135
136				136
137				137
138				138
139				139
140				140
141	000695E	WRITE(44)A(1)		141
142	000696E	WRITE(44)A(1)		142
143	000697E	WRITE(44)A(1)		143
144				144
145	000698E	WRITE(44)A(1)		145
146	000699E	WRITE(44)A(1)		146
147	000700E	WRITE(44)A(1)		147
148	000701E	WRITE(44)A(1)		148
149	000702E	WRITE(44)A(1)		149
150				150
151	000703E	WRITE(44)A(1)		151
152	000704E	WRITE(44)A(1)		152
153	000705E	WRITE(44)A(1)		153
154	000706E	WRITE(44)A(1)		154
155	000707E	WRITE(44)A(1)		155
156	000708E	WRITE(44)A(1)		156
157	000709E	WRITE(44)A(1)		157
158	000710E	WRITE(44)A(1)		158
159	000711E	WRITE(44)A(1)		159
160	000712E	WRITE(44)A(1)		160
161				161
162				162
163				163
164				164
165				165
166	000713E	WRITE(44)A(1)		166
167	000714E	WRITE(44)A(1)		167
168	000715E	WRITE(44)A(1)		168
169				169
170				170
171	000716E	WRITE(44)A(1)		171

Table A.2--Continued

FORWARD	UNIT	STATE	DATE	PAID	AMOUNT	DATE	PAID	AMOUNT
172	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
173	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
174	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
175	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
176	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
177	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
178	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
179	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
180	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
181	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
182	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
183	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
184	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
185	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
186	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
187	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
188	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
189	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
190	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
191	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
192	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
193	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
194	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
195	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
196	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
197	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
198	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
199	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
200	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
201	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
202	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
203	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
204	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
205	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
206	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000
207	0000000	C	00/00/00	0000000	0000000	00/00/00	0000000	0000000

NO. FROM THE GROUP AT THE TIME OF THE PAYMENT. THE DATE OF THE PAYMENT IS SHOWN IN THE DATE COLUMN. THE AMOUNT OF THE PAYMENT IS SHOWN IN THE AMOUNT COLUMN. THE STATE OF THE PAYMENT IS SHOWN IN THE STATE COLUMN. THE UNIT OF THE PAYMENT IS SHOWN IN THE UNIT COLUMN.

Table A.3

LISTING OF SUBROUTINE BCFUNC

LINE NO.	FORTRAN STATEMENT	09/07/82	PROGRAM	PAGE	OF
1	0000001		SUBROUTINE BCFUNC (I, J, K, L, M, N, O, P, Q, R, S, T, U, V, W, X, Y, Z, AA, AB, AC, AD, AE, AF, AG, AH, AI, AJ, AK, AL, AM, AN, AO, AP, AQ, AR, AS, AT, AU, AV, AW, AX, AY, AZ, BA, BB, BC, BD, BE, BF, BG, BH, BI, BJ, BK, BL, BM, BN, BO, BP, BQ, BR, BS, BT, BU, BV, BW, BX, BY, BZ, CA, CB, CC, CD, CE, CF, CG, CH, CI, CJ, CK, CL, CM, CN, CO, CP, CQ, CR, CS, CT, CU, CV, CW, CX, CY, CZ, DA, DB, DC, DD, DE, DF, DG, DH, DI, DJ, DK, DL, DM, DN, DO, DP, DQ, DR, DS, DT, DU, DV, DW, DX, DY, DZ, EA, EB, EC, ED, EE, EF, EG, EH, EI, EJ, EK, EL, EM, EN, EO, EP, EQ, ER, ES, ET, EU, EV, EW, EX, EY, EZ, FA, FB, FC, FD, FE, FF, FG, FH, FI, FJ, FK, FL, FM, FN, FO, FP, FQ, FR, FS, FT, FU, FV, FW, FX, FY, FZ, GA, GB, GC, GD, GE, GF, GG, GH, GI, GJ, GK, GL, GM, GN, GO, GP, GQ, GR, GS, GT, GU, GV, GW, GX, GY, GZ, HA, HB, HC, HD, HE, HF, HG, HH, HI, HJ, HK, HL, HM, HN, HO, HP, HQ, HR, HS, HT, HU, HV, HW, HX, HY, HZ, IA, IB, IC, ID, IE, IF, IG, IH, II, IJ, IK, IL, IM, IN, IO, IP, IQ, IR, IS, IT, IU, IV, IW, IX, IY, IZ, JA, JB, JC, JD, JE, JF, JG, JH, JI, JJ, JK, JL, JM, JN, JO, JP, JQ, JR, JS, JT, JU, JV, JW, JX, JY, JZ, KA, KB, KC, KD, KE, KF, KG, KH, KI, KJ, KK, KL, KM, KN, KO, KP, KQ, KR, KS, KT, KU, KV, KW, KX, KY, KZ, LA, LB, LC, LD, LE, LF, LG, LH, LI, LJ, LK, LL, LM, LN, LO, LP, LQ, LR, LS, LT, LU, LV, LW, LX, LY, LZ, MA, MB, MC, MD, ME, MF, MG, MH, MI, MJ, MK, ML, MM, MN, MO, MP, MQ, MR, MS, MT, MU, MV, MW, MX, MY, MZ, NA, NB, NC, ND, NE, NF, NG, NH, NI, NJ, NK, NL, NM, NN, NO, NP, NQ, NR, NS, NT, NU, NV, NW, NX, NY, NZ, OA, OB, OC, OD, OE, OF, OG, OH, OI, OJ, OK, OL, OM, ON, OO, OP, OQ, OR, OS, OT, OU, OV, OW, OX, OY, OZ, PA, PB, PC, PD, PE, PF, PG, PH, PI, PJ, PK, PL, PM, PN, PO, PP, PQ, PR, PS, PT, PU, PV, PW, PX, PY, PZ, QA, QB, QC, QD, QE, QF, QG, QH, QI, QJ, QK, QL, QM, QN, QO, QP, QQ, QR, QS, QT, QU, QV, QW, QX, QY, QZ, RA, RB, RC, RD, RE, RF, RG, RH, RI, RJ, RK, RL, RM, RN, RO, RP, RQ, RR, RS, RT, RU, RV, RW, RX, RY, RZ, SA, SB, SC, SD, SE, SF, SG, SH, SI, SJ, SK, SL, SM, SN, SO, SP, SQ, SR, SS, ST, SU, SV, SW, SX, SY, SZ, TA, TB, TC, TD, TE, TF, TG, TH, TI, TJ, TK, TL, TM, TN, TO, TP, TQ, TR, TS, TT, TU, TV, TW, TX, TY, TZ, UA, UB, UC, UD, UE, UF, UG, UH, UI, UJ, UK, UL, UM, UN, UO, UP, UQ, UR, US, UT, UY, UV, UW, UX, UY, UZ, VA, VB, VC, VD, VE, VF, VG, VH, VI, VJ, VK, VL, VM, VN, VO, VP, VQ, VR, VS, VT, VU, VV, VW, VX, VY, VZ, WA, WB, WC, WD, WE, WF, WG, WH, WI, WJ, WK, WL, WM, WN, WO, WP, WQ, WR, WS, WT, WU, WV, WW, WX, WY, WZ, XA, XB, XC, XD, XE, XF, XG, XH, XI, XJ, XK, XL, XM, XN, XO, XP, XQ, XR, XS, XT, XU, XV, XW, XX, XY, XZ, YA, YB, YC, YD, YE, YF, YG, YH, YI, YJ, YK, YL, YM, YN, YO, YP, YQ, YR, YS, YT, YU, YV, YW, YX, YY, YZ, ZA, ZB, ZC, ZD, ZE, ZF, ZG, ZH, ZI, ZJ, ZK, ZL, ZM, ZN, ZO, ZP, ZQ, ZR, ZS, ZT, ZU, ZV, ZW, ZX, ZY, ZZ, AA, AB, AC, AD, AE, AF, AG, AH, AI, AJ, AK, AL, AM, AN, AO, AP, AQ, AR, AS, AT, AU, AV, AW, AX, AY, AZ, BA, BB, BC, BD, BE, BF, BG, BH, BI, BJ, BK, BL, BM, BN, BO, BP, BQ, BR, BS, BT, BU, BV, BW, BX, BY, BZ, CA, CB, CC, CD, CE, CF, CG, CH, CI, CJ, CK, CL, CM, CN, CO, CP, CQ, CR, CS, CT, CU, CV, CW, CX, CY, CZ, DA, DB, DC, DD, DE, DF, DG, DH, DI, DJ, DK, DL, DM, DN, DO, DP, DQ, DR, DS, DT, DU, DV, DW, DX, DY, DZ, EA, EB, EC, ED, EE, EF, EG, EH, EI, EJ, EK, EL, EM, EN, EO, EP, EQ, ER, ES, ET, EU, EV, EW, EX, EY, EZ, FA, FB, FC, FD, FE, FF, FG, FH, FI, FJ, FK, FL, FM, FN, FO, FP, FQ, FR, FS, FT, FU, FV, FW, FX, FY, FZ, GA, GB, GC, GD, GE, GF, GG, GH, GI, GJ, GK, GL, GM, GN, GO, GP, GQ, GR, GS, GT, GU, GV, GW, GX, GY, GZ, HA, HB, HC, HD, HE, HF, HG, HH, HI, HJ, HK, HL, HM, HN, HO, HP, HQ, HR, HS, HT, HU, HV, HW, HX, HY, HZ, IA, IB, IC, ID, IE, IF, IG, IH, II, IJ, IK, IL, IM, IN, IO, IP, IQ, IR, IS, IT, IU, IV, IW, IX, IY, IZ, JA, JB, JC, JD, JE, JF, JG, JH, JI, JJ, JK, JL, JM, JN, JO, JP, JQ, JR, JS, JT, JU, JV, JW, JX, JY, JZ, KA, KB, KC, KD, KE, KF, KG, KH, KI, KJ, KK, KL, KM, KN, KO, KP, KQ, KR, KS, KT, KU, KV, KW, KX, KY, KZ, LA, LB, LC, LD, LE, LF, LG, LH, LI, LJ, LK, LL, LM, LN, LO, LP, LQ, LR, LS, LT, LU, LV, LW, LX, LY, LZ, MA, MB, MC, MD, ME, MF, MG, MH, MI, MJ, MK, ML, MM, MN, MO, MP, MQ, MR, MS, MT, MU, MV, MW, MX, MY, MZ, NA, NB, NC, ND, NE, NF, NG, NH, NI, NJ, NK, NL, NM, NN, NO, NP, NQ, NR, NS, NT, NU, NV, NW, NX, NY, NZ, OA, OB, OC, OD, OE, OF, OG, OH, OI, OJ, OK, OL, OM, ON, OO, OP, OQ, OR, OS, OT, OU, OV, OW, OX, OY, OZ, PA, PB, PC, PD, PE, PF, PG, PH, PI, PJ, PK, PL, PM, PN, PO, PP, PQ, PR, PS, PT, PU, PV, PW, PX, PY, PZ, QA, QB, QC, QD, QE, QF, QG, QH, QI, QJ, QK, QL, QM, QN, QO, QP, QQ, QR, QS, QT, QU, QV, QW, QX, QY, QZ, RA, RB, RC, RD, RE, RF, RG, RH, RI, RJ, RK, RL, RM, RN, RO, RP, RQ, RR, RS, RT, RU, RV, RW, RX, RY, RZ, SA, SB, SC, SD, SE, SF, SG, SH, SI, SJ, SK, SL, SM, SN, SO, SP, SQ, SR, SS, ST, SU, SV, SW, SX, SY, SZ, TA, TB, TC, TD, TE, TF, TG, TH, TI, TJ, TK, TL, TM, TN, TO, TP, TQ, TR, TS, TT, TU, TV, TW, TX, TY, TZ, UA, UB, UC, UD, UE, UF, UG, UH, UI, UJ, UK, UL, UM, UN, UO, UP, UQ, UR, US, UT, UY, UV, UW, UX, UY, UZ, VA, VB, VC, VD, VE, VF, VG, VH, VI, VJ, VK, VL, VM, VN, VO, VP, VQ, VR, VS, VT, VU, VV, VW, VX, VY, VZ, WA, WB, WC, WD, WE, WF, WG, WH, WI, WJ, WK, WL, WM, WN, WO, WP, WQ, WR, WS, WT, WU, WV, WW, WX, WY, WZ, XA, XB, XC, XD, XE, XF, XG, XH, XI, XJ, XK, XL, XM, XN, XO, XP, XQ, XR, XS, XT, XU, XV, XW, XX, XY, XZ, YA, YB, YC, YD, YE, YF, YG, YH, YI, YJ, YK, YL, YM, YN, YO, YP, YQ, YR, YS, YT, YU, YV, YW, YX, YY, YZ, ZA, ZB, ZC, ZD, ZE, ZF, ZG, ZH, ZI, ZJ, ZK, ZL, ZM, ZN, ZO, ZP, ZQ, ZR, ZS, ZT, ZU, ZV, ZW, ZX, ZY, ZZ	000	
2					001
3					002
4					003
5					004
6					005
7					006
8					007
9					008
10					009
11					010
12					011
13					012
14					013
15					014
16					015
17					016
18					017
19					018
20					019
21					020
22					021
23					022
24					023
25					024
26					025
27					026
28					027
29					028
30					029
31					030
32					031
33					032
34					033
35					034
36					035
37					036
38					037
39					038
40					039
41					040
42					041
43					042
44					043
45					044
46					045
47					046
48					047
49					048
50					049
51					050
52					051
53					052
54					053
55					054
56					055
57					056

Table A.3--Continued

FORTRAN-770 NAME	FORTRAN-770 STATEMENT	FORTRAN-770 NUMBER
115	CONTINUE	121
116	DO 100 I=1,N	122
117	CONTINUE	123
118	DO 100 I=1,N	124
119	CONTINUE	125
120	DO 100 I=1,N	126
121	CONTINUE	127
122	DO 100 I=1,N	128
123	CONTINUE	129
124	DO 100 I=1,N	130
125	CONTINUE	131
126	DO 100 I=1,N	132
127	CONTINUE	133
128	DO 100 I=1,N	134
129	CONTINUE	135
130	DO 100 I=1,N	136
131	CONTINUE	137
132	DO 100 I=1,N	138
133	CONTINUE	139
134	DO 100 I=1,N	140
135	CONTINUE	141
136	DO 100 I=1,N	142
137	CONTINUE	143
138	DO 100 I=1,N	144
139	CONTINUE	145

NO ERRORS IN STATEMENTS. STATEMENTS 100-145 ARE THE STATEMENTS OF THE PROGRAM. STATEMENTS 146-150 ARE THE STATEMENTS OF THE SUBROUTINE. STATEMENTS 151-155 ARE THE STATEMENTS OF THE FUNCTION. STATEMENTS 156-160 ARE THE STATEMENTS OF THE COMMON BLOCK. STATEMENTS 161-165 ARE THE STATEMENTS OF THE DATA STATEMENT. STATEMENTS 166-170 ARE THE STATEMENTS OF THE END STATEMENT.

Table A.4

LISTING OF SUBROUTINE SPRINT

FORTRAN-VIDE PAGE-NUM	Subroutine Name	09/20/70	11/04/76	PAGE	17
1	000001			000	
2	000002			001	
3	000003			002	
4	000004			003	
5	000005			004	
6	000006			005	
7	000007			006	
8	000008			007	
9	000009			008	
10	000010			009	
11	000011			010	
12	000012			011	
13	000013			012	
14	000014			013	
15	000015			014	
16	000016			015	
17	000017			016	
18	000018			017	
19	000019			018	
20	000020			019	
21	000021			020	
22	000022			021	
23	000023			022	
24	000024			023	
25	000025			024	
26	000026			025	
27	000027			026	
28	000028			027	
29	000029			028	
30	000030			029	
31	000031			030	
32	000032			031	
33	000033			032	
34	000034			033	
35	000035			034	
36	000036			035	
37	000037			036	
38	000038			037	
39	000039			038	
40	000040			039	
41	000041			040	
42	000042			041	
43	000043			042	
44	000044			043	
45	000045			044	
46	000046			045	
47	000047			046	
48	000048			047	
49	000049			048	
50	000050			049	
51	000051			050	
52	000052			051	
53	000053			052	
54	000054			053	
55	000055			054	
56	000056			055	
57	000057			056	

```

FORTRAN VIDE LISTED MATERIALS ARE MADE AVAILABLE TO YOU
***, SEE YOUR BUREAU PACKAGE, 00-101090.
Subroutine Name
1 000001
2
3
4 000004
5
6
7
8
9
10
11
12
13 000013
14
15
16
17
18
19 000019
20
21 000021
22 000022
23 000023
24 000024
25 000025
26 000026
27
28 000028
29 000029
30 000030
31 000031
32 000032
33 000033
34 000034
35 000035
36 000036
37 000037
38 000038
39
40 000040
41 000041
42 000042
43 000043
44
45 000045
46
47
48
49
50
51 000051
52 000052
53 000053
54 000054
55
56 000056
57 000057

```

Table A.4--Continued

FORTMAN-VTID 404-00		08/20/72 11:04:06 PAGE 27	
FORTMAN VTIID: LICENSED VEHICLES AS CLASSIFIED BY I-0000			
*** SEE INPUT TABLES, PACKAGES, 09-101-00.			
58	0004061	1	005
59	0004061	100	100
60	0004281	100	005
61	0004301	100	006
62	0004321	100	107

NO. 0000000000 004-00 SUBMITTED BY 08/20/72 11:04:06 TABLE SPACE 4 KB
 STATEMENT NUMBER: 20 100 8/20/72 11:04:06 STATE SPACE: 121 0000
 SINGLE EXECUTION PLAN: 08/20/72 11:04:06

Table A.5--Continued

FORTRAN-NAME	DESCRIPTION	STATE	CLASS	DATE	PRICE
58 0017801	GADVERSADY			08/20/62	113.00
59 0017821	HYPERADYDZ			08/20/62	113.00
60 0017841	CORRECTION			08/20/62	113.00
61 0017861	FEELING (202) (000126,3)			08/20/62	113.00
62 0017881	FEELING (202) (000126,3)			08/20/62	113.00
63 0017901	FEELING (202) (000126,3)			08/20/62	113.00
64 0017921	FEELING (202) (000126,3)			08/20/62	113.00
65 0017941	FEELING (202) (000126,3)			08/20/62	113.00
66 0017961	FEELING (202) (000126,3)			08/20/62	113.00
67 0017981	FEELING (202) (000126,3)			08/20/62	113.00
68 0018001	FEELING (202) (000126,3)			08/20/62	113.00
69 0018021	FEELING (202) (000126,3)			08/20/62	113.00
70 0018041	FEELING (202) (000126,3)			08/20/62	113.00
71 0018061	FEELING (202) (000126,3)			08/20/62	113.00
72 0018081	FEELING (202) (000126,3)			08/20/62	113.00
73 0018101	FEELING (202) (000126,3)			08/20/62	113.00
74 0018121	FEELING (202) (000126,3)			08/20/62	113.00
75 0018141	FEELING (202) (000126,3)			08/20/62	113.00
76 0018161	FEELING (202) (000126,3)			08/20/62	113.00
77 0018181	FEELING (202) (000126,3)			08/20/62	113.00
78 0018201	FEELING (202) (000126,3)			08/20/62	113.00
79 0018221	FEELING (202) (000126,3)			08/20/62	113.00
80 0018241	FEELING (202) (000126,3)			08/20/62	113.00
81 0018261	FEELING (202) (000126,3)			08/20/62	113.00
82 0018281	FEELING (202) (000126,3)			08/20/62	113.00
83 0018301	FEELING (202) (000126,3)			08/20/62	113.00
84 0018321	FEELING (202) (000126,3)			08/20/62	113.00
85 0018341	FEELING (202) (000126,3)			08/20/62	113.00
86 0018361	FEELING (202) (000126,3)			08/20/62	113.00
87 0018381	FEELING (202) (000126,3)			08/20/62	113.00
88 0018401	FEELING (202) (000126,3)			08/20/62	113.00
89 0018421	FEELING (202) (000126,3)			08/20/62	113.00
90 0018441	FEELING (202) (000126,3)			08/20/62	113.00
91 0018461	FEELING (202) (000126,3)			08/20/62	113.00
92 0018481	FEELING (202) (000126,3)			08/20/62	113.00
93 0018501	FEELING (202) (000126,3)			08/20/62	113.00
94 0018521	FEELING (202) (000126,3)			08/20/62	113.00
95 0018541	FEELING (202) (000126,3)			08/20/62	113.00
96 0018561	FEELING (202) (000126,3)			08/20/62	113.00
97 0018581	FEELING (202) (000126,3)			08/20/62	113.00
98 0018601	FEELING (202) (000126,3)			08/20/62	113.00
99 0018621	FEELING (202) (000126,3)			08/20/62	113.00
100 0018641	FEELING (202) (000126,3)			08/20/62	113.00
101 0018661	FEELING (202) (000126,3)			08/20/62	113.00
102 0018681	FEELING (202) (000126,3)			08/20/62	113.00
103 0018701	FEELING (202) (000126,3)			08/20/62	113.00
104 0018721	FEELING (202) (000126,3)			08/20/62	113.00
105 0018741	FEELING (202) (000126,3)			08/20/62	113.00
106 0018761	FEELING (202) (000126,3)			08/20/62	113.00
107 0018781	FEELING (202) (000126,3)			08/20/62	113.00
108 0018801	FEELING (202) (000126,3)			08/20/62	113.00
109 0018821	FEELING (202) (000126,3)			08/20/62	113.00
110 0018841	FEELING (202) (000126,3)			08/20/62	113.00
111 0018861	FEELING (202) (000126,3)			08/20/62	113.00
112 0018881	FEELING (202) (000126,3)			08/20/62	113.00
113 0018901	FEELING (202) (000126,3)			08/20/62	113.00
114 0018921	FEELING (202) (000126,3)			08/20/62	113.00

Table A.5--Continued

PORTMAN-VIIID 008-00	PORTMAN VTIID: CUFUNSHO WENSTIPITRE WIGHTS AS STABLE I CUFUNSHO WENSTIPITRE WIGHTS AS STABLE I	09/20/92 11:20:06 PMG M 12
115		522
116		523
117		524
118		525
119		526
120		527
121		528
122		529
123		530
124		531
125		532
126		533
127		534
128		535
129		536
130		537
131		538
132		539
133		540
134		541
135		542
136		543
137		544
138		545
139		546
140		547
141		548
142		549
143		550
144		551
145		552
146		553
147		554
148		555
149		556
150		557
151		558
152		559
153		560
154		561
155		562
156		563
157		564
158		565
159		566
160		567
161		568
162		569
163		570
164		571
165		572
166		573
167		574
168		575
169		576
170		577
171		578

Table A.5--Continued

FORTRAN STATEMENT	FORTRAN STATEMENT	FORTRAN STATEMENT	FORTRAN STATEMENT	FORTRAN STATEMENT	FORTRAN STATEMENT
172 00224E1	074120,0	00224E1	074120,0	00224E1	074120,0
173 00224E2	074120,0	00224E2	074120,0	00224E2	074120,0
174 00224E3	074120,0	00224E3	074120,0	00224E3	074120,0
175 00224E4	074120,0	00224E4	074120,0	00224E4	074120,0
176 00224E5	074120,0	00224E5	074120,0	00224E5	074120,0
177 00224E6	074120,0	00224E6	074120,0	00224E6	074120,0
178 00224E7	074120,0	00224E7	074120,0	00224E7	074120,0
179 00224E8	074120,0	00224E8	074120,0	00224E8	074120,0
180 00224E9	074120,0	00224E9	074120,0	00224E9	074120,0
181		181		181	
182		182		182	
183		183		183	
184		184		184	
185		185		185	
186		186		186	
187 00224E1	074120,0	187 00224E1	074120,0	187 00224E1	074120,0
188 00224E2	074120,0	188 00224E2	074120,0	188 00224E2	074120,0
189 00224E3	074120,0	189 00224E3	074120,0	189 00224E3	074120,0
190		190		190	
191		191		191	
192		192		192	
193		193		193	
194 00224E1	074120,0	194 00224E1	074120,0	194 00224E1	074120,0
195 00224E2	074120,0	195 00224E2	074120,0	195 00224E2	074120,0
196		196		196	
197		197		197	
198 00224E1	074120,0	198 00224E1	074120,0	198 00224E1	074120,0
199 00224E2	074120,0	199 00224E2	074120,0	199 00224E2	074120,0
200 00224E3	074120,0	200 00224E3	074120,0	200 00224E3	074120,0
201 00224E4	074120,0	201 00224E4	074120,0	201 00224E4	074120,0
202 00224E5	074120,0	202 00224E5	074120,0	202 00224E5	074120,0
203		203		203	
204		204		204	
205		205		205	
206		206		206	
207		207		207	
208		208		208	
209		209		209	
210		210		210	
211 00224E1	074120,0	211 00224E1	074120,0	211 00224E1	074120,0
212 00224E2	074120,0	212 00224E2	074120,0	212 00224E2	074120,0
213		213		213	
214		214		214	
215		215		215	
216		216		216	
217		217		217	
218		218		218	
219		219		219	
220		220		220	
221 00224E1	074120,0	221 00224E1	074120,0	221 00224E1	074120,0
222 00224E2	074120,0	222 00224E2	074120,0	222 00224E2	074120,0
223 00224E3	074120,0	223 00224E3	074120,0	223 00224E3	074120,0
224 00224E4	074120,0	224 00224E4	074120,0	224 00224E4	074120,0
225		225		225	
226 00224E1	074120,0	226 00224E1	074120,0	226 00224E1	074120,0
227		227		227	
228 00224E1	074120,0	228 00224E1	074120,0	228 00224E1	074120,0

Table A.5--Continued

FORTRAN-77 ID	FORTRAN-77 CODE	FORTRAN-77 STATEMENT	FORTRAN-77 STATEMENT	FORTRAN-77 STATEMENT	FORTRAN-77 STATEMENT	FORTRAN-77 STATEMENT
229	0021401	CALL SUBROUTINE (...)				636
230	0021402					637
231	0021403					638
232	0021404					639
233	0021405					640
234	0021406					641
235	0021407					642
236	0021408					643
237	0021409					644
238	0021410					645
239	0021411					646
240	0021412					647
241	0021413					648
242	0021414					649
243	0021415					650
244	0021416					651
245	0021417					652
246	0021418					653
247	0021419					654
248	0021420					655
249	0021421					656
250	0021422					657
251	0021423					658
252	0021424					659
253	0021425					660
254	0021426					661
255	0021427					662
256	0021428					663
257	0021429					664
258	0021430					665
259	0021431					666
260	0021432					667
261	0021433					668
262	0021434					669
263	0021435					670
264	0021436					671
265	0021437					672
266	0021438					673
267	0021439					674
268	0021440					675
269	0021441					676
270	0021442					677
271	0021443					678
272	0021444					679
273	0021445					680
274	0021446					681
275	0021447					682
276	0021448					683
277	0021449					684
278	0021450					685
279	0021451					686
280	0021452					687
281	0021453					688
282	0021454					689
283	0021455					690
284	0021456					691
285	0021457					692

Table A.5--Continued

FORTRAN-77 ID	FORTRAN-77 CODE	FORTRAN-77 NAME	DESCRIPTION	SYMBOLIC NAME	ADDRESS	PAGE
286	003CC21		FUNCTION		693	
287	003CC21		FUNCTION		694	
288	003CC21		FUNCTION		695	
289	003CC21		FUNCTION		696	
290	003CC21		FUNCTION		697	
291	003CC21		FUNCTION		698	
292	003CC21		FUNCTION		699	
293	003CC21		FUNCTION		700	
294	003CC21		FUNCTION		701	
295	003CC21		FUNCTION		702	
296	003CC21		FUNCTION		703	
297	003CC21		FUNCTION		704	
298	003CC21		FUNCTION		705	
299	003CC21		FUNCTION		706	
300	003CC21		FUNCTION		707	
301	003CC21		FUNCTION		708	
302	003CC21		FUNCTION		709	
303	003CC21		FUNCTION		710	
304	003CC21		FUNCTION		711	
305	003CC21		FUNCTION		712	
306	003CC21		FUNCTION		713	
307	003CC21		FUNCTION		714	
308	003CC21		FUNCTION		715	
309	003CC21		FUNCTION		716	
310	003CC21		FUNCTION		717	
311	003CC21		FUNCTION		718	
312	003CC21		FUNCTION		719	
313	003CC21		FUNCTION		720	
314	003CC21		FUNCTION		721	
315	003CC21		FUNCTION		722	
316	003CC21		FUNCTION		723	
317	003CC21		FUNCTION		724	
318	003CC21		FUNCTION		725	
319	003CC21		FUNCTION		726	
320	003CC21		FUNCTION		727	
321	003CC21		FUNCTION		728	
322	003CC21		FUNCTION		729	
323	003CC21		FUNCTION		730	
324	003CC21		FUNCTION		731	
325	003CC21		FUNCTION		732	
326	003CC21		FUNCTION		733	
327	003CC21		FUNCTION		734	
328	003CC21		FUNCTION		735	
329	003CC21		FUNCTION		736	
330	003CC21		FUNCTION		737	
331	003CC21		FUNCTION		738	
332	003CC21		FUNCTION		739	
333	003CC21		FUNCTION		740	
334	003CC21		FUNCTION		741	
335	003CC21		FUNCTION		742	
336	003CC21		FUNCTION		743	
337	003CC21		FUNCTION		744	
338	003CC21		FUNCTION		745	
339	003CC21		FUNCTION		746	
340	003CC21		FUNCTION		747	
341	003CC21		FUNCTION		748	
342	003CC21		FUNCTION		749	

Table A.5--Continued

FORTMAN-VTIO 000-00		09/20/99	13/00/06	page	77	16
FORTMAN, VINDI LICENSIO RESEPTIO SIBIS AS STABIT F. LIC USE 1-0036		***, SEE OTHER STATING PACKAGE, 00-001-009.				
343	004194I	77720,0				750
344	004194I	FF(1,2,3,4000) GOTO 500				751
345	004194I	WRITE(10)100,500				752
346	004194I	FORALL(1,2,3,4) DO				753
347	004194I	500 500 1000,500				754
348	004194I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				755
349	004202I	1				756
350	004202I	FORALL(1,2,3,4) DO				757
351	004202I	500 500 1000,500				758
352	004202I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				759
353	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				760
354	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				761
355	004306I	1				762
356	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				763
357	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				764
358	004306I	1				765
359	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				766
360	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				767
361	004306I	1				768
362	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				769
363	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				770
364	004306I	1				771
365	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				772
366	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				773
367	004306I	1				774
368	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				775
369	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				776
370	004306I	1				777
371	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				778
372	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				779
373	004306I	1				780
374	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				781
375	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				782
376	004306I	1				783
377	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				784
378	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				785
379	004306I	1				786
380	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				787
381	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				788
382	004306I	1				789
383	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				790
384	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				791
385	004306I	1				792
386	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				793
387	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				794
388	004306I	1				795
389	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				796
390	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				797
391	004306I	1				798
392	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				799
393	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				800
394	004306I	1				801
395	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				802
396	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				803
397	004306I	1				804
398	004306I	004306I=FF(1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20)				805
399	004306I	WRITE(10)1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20				806

Table A.5--Continued

FORTRAN-VIEW RUN-NO.	FORTRAN VIEW SUBJECTS - DATES AS SHOWN IN SUBJECTS LISTING	09/20/52	11/03/52	04/10/53	04/17/53
400	0006781	IF(1,10,1) STOP			407
401	0006781	IF(1,10,1) STOP			408
402	0006781	IF(1,10,1) STOP			409
403	0006781	IF(1,10,1) STOP			410
404	0006781	IF(1,10,1) STOP			411
405	0006781	IF(1,10,1) STOP			412
406	0006781	IF(1,10,1) STOP			413
407	0006781	IF(1,10,1) STOP			414
408	0006781	IF(1,10,1) STOP			415
409	0006781	IF(1,10,1) STOP			416
410	0006781	IF(1,10,1) STOP			417
411	0006781	IF(1,10,1) STOP			418
412	0006781	IF(1,10,1) STOP			419
413	0006781	IF(1,10,1) STOP			420
414	0006781	IF(1,10,1) STOP			421
415	0006781	IF(1,10,1) STOP			422
416	0006781	IF(1,10,1) STOP			423
417	0006781	IF(1,10,1) STOP			424
418	0006781	IF(1,10,1) STOP			425
419	0006781	IF(1,10,1) STOP			426
420	0006781	IF(1,10,1) STOP			427
421	0006781	IF(1,10,1) STOP			428
422	0006781	IF(1,10,1) STOP			429
423	0006781	IF(1,10,1) STOP			430
424	0006781	IF(1,10,1) STOP			431
425	0006781	IF(1,10,1) STOP			432
426	0006781	IF(1,10,1) STOP			433
427	0006781	IF(1,10,1) STOP			434
428	0006781	IF(1,10,1) STOP			435
429	0006781	IF(1,10,1) STOP			436
430	0006781	IF(1,10,1) STOP			437
431	0006781	IF(1,10,1) STOP			438
432	0006781	IF(1,10,1) STOP			439
433	0006781	IF(1,10,1) STOP			440
434	0006781	IF(1,10,1) STOP			441
435	0006781	IF(1,10,1) STOP			442
436	0006781	IF(1,10,1) STOP			443
437	0006781	IF(1,10,1) STOP			444
438	0006781	IF(1,10,1) STOP			445
439	0006781	IF(1,10,1) STOP			446
440	0006781	IF(1,10,1) STOP			447
441	0006781	IF(1,10,1) STOP			448

NO. EMPLOYEES 403-500 SUBJECTS 403-500 DATES 09/20/52 11/03/52 04/10/53 04/17/53
 STATEMENT DIFFERS TO THE SUBJECTS LISTING - STATE SUBJECTS LISTING
 SINGLE PRECISHP FLOATING OF SUBJECTS LISTING FOR 11/03/52

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ANALYSIS OF THE LARGE URBAN FIRE ENVIRONMENT
Part II. Parametric Analysis and Model City Simulations
Unclassified

D. A. Larson and R. D. Small
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12340 Santa Monica Boulevard, Los Angeles, California 90025

PSR Report 1210, November 1982, 120 pp., Contract ERM-C-0747, Work Unit 2564E

This report considers the fire environment that would result from a megaton-yield nuclear weapon explosion. An analysis (developed in Part I) that treats the physics of the burning zone and the volume immediately above it (turning region) is used to predict the velocity, temperature, and pressure fields of large area fires.

A sensitivity study explores the influence of turbulence, radiation, fire size, and burning intensity on the mean temperature levels and velocity fields. The results show hurricane-force velocities developing as the fire size or burning rate is increased. A sample calculation illustrates the change in fire-wind velocities as the fire evolves over time.

Calculations of the burning region for three model urban areas show the influence of building density and urban sprawl on the resulting fire environment. An additional set of predictions accounts for reduction of the fire intensity by blast in the urban center. For the latter cases, the temperature distribution is changed markedly, though the magnitude of the induced fire winds is not appreciably reduced.

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Part II. Parametric Analysis and Model City Simulations

Summary

By
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R. D. Small

November 1982

Final Report
Contract EMW-C-0747, Work Unit 2564E

For
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National Preparedness Programs
Washington, D.C. 20472

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SUMMARY

This report considers the large-fire environment that would occur in an urban area subject to a nuclear weapon explosion. The effects of system parameters are explored in a sensitivity study, and results for three model cities are presented.

The examples are characterized by extensive areas simultaneously burning, strong buoyancy, and large temperature gradients. Several such fires occurred during World War II. Though those fires were dramatic in intensity and destructiveness, each involved a relatively small area. A nuclear weapon explosion could generate a far larger area fire and a more severe fire environment. This report is intended to define such large fires. The results should be applicable for damage evaluation, formulation of shelter requirements, and rescue planning.

The calculations are based on the theory developed in Part I of this report, which is applicable to the fire zone and the volume immediately above it (turning region). The effects of variable area heating, turbulence, strong buoyancy, large temperature changes, and radiation are treated. The induced fire winds and rapid temperature changes at the fire periphery are uniquely determined by the use of jump conditions. Simulations of the Hamburg firestorm and a large Flambeau fire agreed well with available data.

The parametric analysis considers a large area fire and the effects of fire size, heating rates, mixing coefficients, and hot gas/smoke radiation. The results show the influence of those variables on the induced fire winds, mean temperature, and pressure gradients. In general, an increase in either the fire size or heating rate raises the mean temperature levels and the induced fire-wind velocities. For the larger heat release rates or fire sizes, the attendant increases in mean temperature and velocity are limited by compressibility effects.

Fires such as may result from a megaton-yield explosion are analyzed for three model urban areas. Each city is characterized by

a high-density center, a surrounding belt of mixed residential/ industrial construction, and a lower density suburban belt. Each model city portrays a different degree of building density and urban sprawl. The results illustrate how a particular city geometry affects the velocity and temperature fields for a given fire. An additional series of computations considers reduction of the fire area by severe blast damage and debris formation. For those calculations, complete burning was allowed in an annular area, with the fire intensity significantly reduced in the center.

Finally, the model is employed to estimate the behavior of the velocity and temperature fields as a function of fire evolution. Those calculations may indicate the most appropriate periods for effecting rescue operations as well as provide an estimate of time-dependent shelter loadings.