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A PROBLEM IN NUCLEAR THERMAL RADIATION ENVIRONMENT SIMULATION FOR SYSTEM SURVIVABILITY

> Ennis F. Quigley John M. Evans

> > August 1980





US ARMY ARMAMENT RESEARCH AND DEVELOPMENT COMMAND BALLISTIC RESEARCH LABORATORY ABERDEEN PROVING GROUND, MARYLAND

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Thermal Flux	
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A series of experiments was conducted at two nucl	ear thermal radiation sim-
ulators; the White Sands Missile Range Solar Furnac	e facility and the Wright-
racterson Air Force Base Quartz Lamps facility. The	e purpose of these experi-
ments was to study the cifect of similator radiation the surfaces. The surfaces absorbed by painted surfaces.	on spectrum on the amount of
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thermal radiation spectrum. Thirty-nine percent me	magning dependent on the
the solar furnace than was absorbed using the quart	z lamps. The experimental
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results also indicated that the amount of absorbed energy may be independent of of thermal radiation flux.

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I. INTRODUCTION

Until the middle of 1977 the survivability testing of full scale US Army tactical systems to simulated nuclear thermal environments and to combined nuclear thermal-blast environments was not possible. Since that time Scientific Applications, Inc., under contract to the Defense Nuclear Agency, has been developing a general purpose thermal radiation simulator¹ which may provide the means for such testing. This simulator is transportable and self-consuming and can be used to irradiate targets ranging in sizes from laboratory models to full-size prototypes. The thermal radiation is produced by the burning of aluminum powder in an oxygen atmosphere at a temperature of approximately 3600° K. This simulator has been used by the Ballistic Research Laboratory for nuclear thermal survivability studies of several Lance Missile System components and for combined nuclear thermal-blast survivability studies of C^{2} systems. Since there now exists the possibility to conduct thermal and thermal-blast survivability testings of large tactical systems, it is necessary to determine those characteristics of the nuclear thermal radiation environment which must be simulated in order to obtain valid results from such tests.

The effects of the thermal environment on a system are due to the absorption of all or part of the radiant energy incident on the exposed surfaces. These surfaces are generally painted and the amount of energy absorbed by the system is highly dependent on the thermal radiation absorptivity of the surfaces. To determine to what extent the nuclear thermal environment must be simulated, one must know the characteristics of the environment at the target and the dependence of the thermal absorptivity on these characteristics.

The general characteristics of the nuclear thermal environment at a tactical target are the pulse shape, the rise time of the pulse, the maximum thermal flux, the total thermal fluence, and the time dependent radiation spectrum. Of these five characteristics, only the first four are considered for survivability testing. The fifth characteristic, the radiation spectrum, is never considered because of the inherent difficulties associated with spectral characterization. Consequently, the most difficult characteristic to simulate is then the radiation spectrum. To study the effect of simulator spectrum on the amount of thermal energy absorbed by painted surfaces, a series of experiments have been conducted at two nuclear thermal radiation environment simulators; the White Sands

¹J. F. Dishon, "Large Scale Thermal Radiation Simulator", DNA 001-77-C-0206, 12 May 1977, 1st Monthly Report.

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Missile Range (WSMR) Solar Furnace facility² and the Wright-Patterson Air Force Base(WPAFB) Quartz Lamp facility³. This report describes these experiments and their results.

II. PROCEDURE

Assuming that a plate is thermally thin, the amount of thermal energy absorbed by the painted surface is directly proportional to the time derivative of the plate temperature. This functional relationship is:

$$\alpha Q = pc1 \frac{dT}{dt} + H$$
(1)

where

\$: # (j)

α	<pre>= surface absorptivity</pre>
Q	= thermal flux
р	= plate density
c	= plate specific heat
1	= plate thickness
Т	= plate temperature
t	= time
Н	= thermal energy losses by the plate

If an effective absorptivity, α_{e} , is defined such that

and

$$\alpha_{e} \dot{Q} = pc1 \frac{dT}{dt}$$
(2)

the difficulties of determining H are avoided. By obtaining α_e for each simulator, one can determine the effect of simulator spectrum on the amount of energy absorbed by painted surfaces.

²White Sands Solar Facility Experimenter's Guide, 1977.

³A. Servois, J. Olson, and H. Hilt, "Tri-Service Thermal Flash Test Facility", DNA 44 882, March 1978, University of Dayton, Dayton, OH 4509 (AD-A056 321)

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The paint-primer samples which were tested are listed in Table 1.

Sample	Primer	Paint	
A .	TT-P-636*	MIL-E-52798A*	
В	TT-P-664	M1L-E-52835A	
С	TT-P-664	MIL-L-52909	
D	TT-P-664	M1L-L-52926	
Е	TT-P-664	M1L-L-52929	

TABLE 1. Forest Green Camouflage Paint-Primer Samples

The substrate for each sample was a 50 mm x 50 mm x 1.0 mm ASIS 1020 cold rolled steel plate and the samples were fabricated by the US Army Camouflage Laboratory at Fort Belvoir, VA. One end of a one meter length K-type thermocouple was spot-welded at the center of the back of ϵ ach plate.

As previously mentioned, the thermal simulators used in the tests were the Quartz Lamp (QL) facility at WPAFB and the Solar Furnace (SF) facility at WSMR. The thermal environment characteristics of the QL were a radiating temperature of approximately 2800°K and a thermal flux which was uniform over the entire surface of the sample. The SF had a radiating temperature of approximately 6000°K and a thermal flux distribution over the sample surface as shown in Figure 1. Calculations were made to determine the effect of a nonuniform, but axial symmetric, flux distribution on the back center temperature of the plate. The results of the calculations showed that the differences between temperatures produced by a uniform flux distribution and the SF flux distribution were less than 1° for the flux values used in the tests. The thermal pulse characteristics for the tests are given in Table 2.

TABLE 2. Simulator Pulse Characteristics

Characteristic	Quartz Lamps	Solar Furnace
Pulse Shape	Rectangular	Rectangular
Thermal Flux	0.84 MW/m ² 1.59 MW/m ²	0.84 MW/m^{2} 1.63 MW/m ²
Thermal Fluence Rise Time	2.51 MJ/m^2 t < 30 ms	2.51 MJ/m^2 t < 30 ms

*These numbers refer to the Military Specifications of the primers and paints.



Figure 1. Solar Furnace Flux Distribution at Focal Plane

For the QL tests the output of the thermocouples was recorded on a HP 1360 X-Y recorder; for the SF tests, the thermocouple output was recordered on a Gould 816 strip-chart recorder. Three to five specimens of each paint-primer combination were exposed. Measurement of the thermal flux was performed before and after each set of paint-primer combinations for both simulators.

III. RESULTS AND DISCUSSIONS

Figures 2 and 3 are plots of the average plate temperatures, \overline{T} , of all the samples exposed to the QL simulator for thermal flux values of 0.84 MW/m^2 and 1.59 MW/m², respectively. (For all figures involving \overline{T} , thermal fluence, Q, rather than time was used as the independent parameter since Q = Qt.) The plotted data* indicates for each flux that the average temperature response of the plates are approximately the same for each sample and that \overline{T} might be expressable as a linear function of Q. Figures 4 through 8 are plots of the linear regression curve of \overline{T} for each sample. The data and curves in these figures suggest that \overline{T} is independent of the thermal flux value. An effective absorptivity for each sample can be calculated from Equation (2) since $\overline{T} = a+bQ = a+bQt$ and $d\overline{T} = b0$. The $d\overline{t}_{\overline{t}}$ values of p and c used in the calculations are 7.833 x 10³ kg/m⁵ and 465 J/kg°K, respectively. The values of α_e are given in Table 3 where $\overline{\alpha}_e$ is the average effective absorptivity of all the samples for that flux value.

Sample	$Flux = 0.84 \text{ MW/m}^2$	$F1ux = 1.59 \text{ MW/m}^2$
А	0.48	0.43
В	0.47	0.46
С	0.45	0.43
D	0.50	0.49
E	0.45	0.41
άe	0.47 ± 0.02	0.45 ± 0.03

TABLE 3. Effective Absorptivity for Quart: Lamps Source

"See Appendix A for tabulated data.

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Figure 2. Average Temperature Data of all Samples - Quartz Lamps

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Figure 4. Sampe A Average Temperature - Quart: Lamps

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Sample B Average Temperature - Quartz Lamps Figure 5.







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Figure 7. Sample D Average Temperature - Quart: Lamps

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It can be concluded from the test data that the effective absorptivities for all the Forest Green Camouflage paint-primer combination have the same value, can be considered independent of temperature for temperatures less than 300° C, and may be independent of the thermal flux for the QL simulator.

Figures 9 and 10 are plots of \overline{T} for values of Q of all the samples for the SF simulator for thermal flux values of 0.84 MW/m⁻ and 1.63 MW/m⁻, respectively. The data^{*} in Figure 9 indicates for T < 300°C that the average temperature response of the plates for each sample are approximately the same and that \overline{T} could be expressed as a linear function of Q. For \overline{T} > 300°C, samples B, C, and D show similar temperature behavior and samples A and E show similar temperature behavior. Physical examination of samples A and E immediately following exposure revealed that all or most of the paint had been burnt off. The data* in Figure 10 indicates for \overline{T} < 200°C that the average temperature response of each sample is approximately the same and that \overline{T} could be expressed as a linear function of Q. For $T > 200^{\circ}C$, samples B and C show similar temperature behavior and samples A and F show similar temperature behavior. Sample D for this flux value did not behave like sample B and C; but more like samples A and E. The reason for this is not clear. After exposure, samples A and F had all or most of the paint burnt off. Figures 11 through 15 are plots of the regression curve and data for each sample. Each curve ends at the last data point used in obtaining the curve. The data and curves in these figures suggest that \overline{T} is independent of the flux value for all measured temperatures. The data also indicates that for $\overline{T} > 300^{\circ}$ C, the \overline{T} of each sample could be expressed as a linear function of Q. The calculated α_{α} of each sample is given in Table 4 where α_{α} is the average effective absorptivity of all the samples for that flux value.

Sample	$Flux = 0.84 \text{ MW/m}^2$	$Flux = 1.63 \text{ MW/m}^2$
A	0.64	0.62
В	0.67	0.64
С	0.66	0.64
D	0.65	0.61
Е	0.66	0.61
a e	0.66 + 0.01	0.62 ± 0.02

TABLE 4. Effective Absorptivity for Solar Furnace Source

*See Appendix A for tabulated data.



Figure 9. Average Temperature Data of all Samples - Solar Furnace

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Figure 10. Average Temperature Data of all Samples - Solar Furnace

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Figure 12. Sample B Average Temperature - Solar Furnace



Figure 15. Sample C Average Temperature - Solar Furnace





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It can be concluded from the test data that the effective absorptivities for all the Forest Green Camouflage paint-primer combinations have the same value, can be considered independent of temperature for temperatures less than 300°C, and may be independent of the thermal flux for the SF simulator.

Table 5 contains the flux values and effective absorptivity of all the samples for each simulator. The average effective absorptivity of all the samples for both flux values for each simulator is given by $\overline{\alpha}_{\alpha}$

Flux	Quartz Lamps	Solar Furnace
0.84 MW/m ²	0.47	0.66
1.59 MW/m^2	0.45	-
1.63 MW/m^2	-	0.62
ā	0.46 + 0.02	0.64 ± 0.02

TABLE 5. Effective Absorptivity

The average effective absorptivity associated with the Solar Furnace simulator is 39% greater than the average effective absorptivity associated with the Quartz Lamps simulator.

IV. CONCLUSIONS

The amount of thermal energy absorbed by a system whose exposed surfaces are painted with a Forest Green Camouflage paint-primer combination is highly dependent on the thermal radiation spectrum and may be independent of the theremal radiation flux. Thirty-nine percent more energy was absorbed using the solar furnace than was absorbed using the quartz lamps. The implication of the dependence of the absorptivity on the thermal radiation spectrum is obvious. The survivability of a system to a simulated thermal or thermal-blast environment may be simulator dependent. Consequently, until it has been determined to what extent the nuclear thermal radiation environment must be simulated, system survivability results using thermal simulators are questionable.

APPENDIX A. Average Temperature Data

Table Al gives the number of specimens of each paint-primer sample exposed at each simulator.

Sample		Quartz Lamps		Solar Furnace	
	0.84 MW/m ²	1.59 MW/m ²	0.84 MW/m ²	1.63 MW/m ²	
А	5	5	3	5	
В	5	5	4	5	
С	5	5	3	5	
D	5	5	3	4	
E	5	5	4	4	

TABLE A1. Number ot Paint-Primer Exposures

Tables A2 through A5 list the thermal fluence, Q, and the average temperature, \overline{T} , with its standard deviation. The subscript of \overline{T} designates the sample and all temperatures and standard deviation were rounded off to the nearest degree.

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$F1ux = 0.84 \text{ MW/m}^2$					
$Q_{(MJ/m^2)}$	T _A (°C)	T _B (°C)	$\frac{\overline{T}_{C}}{C}$ (°C)	Τ _μ (°C)	T _E (°C)
0.00	30	30	30	30	30
0.17	53 <u>+</u> 1	53 <u>+</u> 1	52 + 1	51 + 1	52 <u>+</u> 1
0.34	75 + 1	76 <u>+</u> 1	74 + 1	72 <u>+</u> 2	74 + 2
0.50	98 + 1	99 + 1	96 ± 1	94 + 1	95 + 2
0.67	119 + 2	122 + 1	118 + 1	116 + 1	117 + 1
0.84	144 + 2	145 - 2	139 + 1	138 ± 1	139 + 2
1.01	167 + 1	168 ± 2	159 + 2	159 + 2	160 + 2
1.17	189 + 1	189 + 1	182 + 2	180 <u>+</u> 1	182 + 3
1.34	210 + 2	212 + 2	202 + 1	203 + 2	203 + 2
1.51	233 + 1	231 + 2	220 + 2	225 + 2	221 + 2
1.68	252 + 2	254 + 1	240 + 2	246 + 1	241 + 2
1.84	271 + 1	274 + 2	259 + 2	272 + 1	261 ± 3
2.01	293 + 2	294 + 2	278 ± 2	297 + 2	279 + 3
2.18	315 + 3	514 + 2	298 ± 3	322 + 3	297 + 3
2.35	336 <u>+</u> 3	334 <u>+</u> 3	316 <u>+</u> 3	349 + 4	317 + 4
2.51	347 + 3	349 + 4	*	*	*

TABLE A2. Average Temperatures for Quartz Lamps

* No data

Flux = 1.5	9 MW/m ²				
$Q (MJ/m^2)$	T _{A (C)}	T _B (C)	<u>T</u> (C)	T _D (C)	T _E (C)
0.00	30	30	30	30	30
0.16	49 + 2	52 <u>+</u> 1	48 <u>+</u> 2	49 <u>+</u> 1	48 + 1
0.32	68 <u>+</u> 3	71 <u>+</u> 1	68 <u>+</u> 2	68 <u>+</u> 2	68 <u>+</u> 1
0.48	88 <u>+</u> 4	92 <u>+</u> 2	87 <u>+</u> 1	87 <u>+</u> 2	88 + 1
0.64	108 + 4	113 <u>+</u> 2	106 <u>+</u> 1	107 + 1	107 + 2
0.80	127 + 4	131 <u>+</u> 1	127 <u>+</u> 2	128 + 2	128 + 2
0.96	148 <u>+</u> 6	153 + 2	145 + 1	148 + 3	148 + 2
1.11	168 + 6	174 <u>+</u> 2	164 + 2	168 + 3	165 + 3
1.27	185 <u>+</u> 7	193 <u>+</u> 1	183 + 2	188 + 3	185 + 2
1.43	206 + 7	214 + 1	201 + 2	209 + 3	205 + 2
1.59	222 + 8	232 + 2	219 + 1	232 + 5	225 + 2
1.75	238 + 9	252 + 2	237 + 2	254 + 4	242 ± 2
1.91	264 <u>+</u> 10	273 + 3	256 + 3	276 + 5	258 + 3
2.07	276 + 12	289 + 2	273 + 3	299 + 6	278 + 3
2.23	290 <u>+</u> 12	308 + 3	291 + 3	321 + 7	 296 + 3
2.39	299 <u>+</u> 12	325 <u>+</u> 4	309 <u>+</u> 4	343 <u>+</u> 8	314 <u>+</u> 4

TABLE A3. Average Temperatures for Quartz Lamps

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F1ux = 0.84	4 MW/m ²				
Q (MJ/m ²)	T _A (C)	T _B (C)	Ť _C (C)	τ _D (C)	Τ _E (C)
0.0	30	30	30	30	30
0.17	62 + 3	63 + 2	61 + 2	62 + 1	63 <u>+</u> 1
0.34	92 + 2	97 + 3	93 + 1	96 + 1	94 + 2
0.50	124 + 2	128 + 3	125 + 1	127 + 1	125 + 5
0.67	154 + 3	160 + 3	156 + 2	158 + 2	156 + 6
0.84	185 + 4	190 + 4	184 + 1	190 + 1	186 + 6
1.01	215 + 3	220 + 3	213 + 1	218 + 4	215 + 6
1.17	238 + 3	248 <u>+</u> 3	242 + 1	243 + 4	244 + 6
1.34	263 + 3	276 + 4	272 + 1	269 <u>+</u> 4	269 + 9
1.51	287 + 3	306 <u>+</u> 6	300 + 2	294 <u>+</u> 5	287 + 12
1.68	307 <u>+</u> 5	326 <u>+</u> 4	328 + 3	319 <u>+</u> 6	306 <u>+</u> 15
1.84	322 + 6	346 <u>+</u> 4	349 <u>+</u> 2	341 + 8	323 <u>+</u> 16
2,01	334 <u>+</u> 7	367 <u>+</u> 5	368 <u>+</u> 2	360 + 9	339 <u>+</u> 19
2.18	346 + 7	386 <u>+</u> 6	387 <u>+</u> 2	380 + 9	355 <u>+</u> 21
2.35	360 + 5	404 <u>+</u> 6	403 + 3	401 + 9	372 <u>+</u> 25
2.51	379 <u>+</u> 5	424 + 8	422 + 1	426 + 9	389 <u>+</u> 30

TABLE A4. Average Temperatures for Solar Furnace

Section States

$F1ux = 1.63 \text{ MW/m}^2$					
<u>Q (MJ/m²)</u>	T _A (C)	T _B (C)	T _C (C)	T _D (C)	T _E (C)
0.00	30	30	30	30	30
0.16	60 <u>+</u> 4	59 <u>+</u> 3	58 <u>+</u> 1	59 <u>+</u> 1	56 <u>+</u> 3
0.33	91 <u>+</u> 4	92 <u>+</u> 3	87 <u>+</u> 2	89 <u>+</u> 2	86 <u>+</u> 6
0.49	120 <u>+</u> 5	121 <u>+</u> 4	117 <u>+</u> 3	120 <u>+</u> 3	114 + 8
0.65	150 <u>+</u> 6	150 <u>+</u> 5	145 <u>+</u> 2	148 <u>+</u> 2	143 <u>+</u> 9
0.82	180 <u>+</u> 7	180 <u>+</u> 5	176 <u>+</u> 4	174 <u>+</u> 3	170 <u>+</u> 11
0.98	205 <u>+</u> 6	210 <u>+</u> 1	205 <u>+</u> 3	195 <u>+</u> 3	197 + 14
1.14	226 <u>+</u> 7	235 <u>+</u> 5	231 <u>+</u> 3	215 + 3	222 <u>+</u> 15
1.31	247 <u>+</u> 8	261 <u>+</u> 7	259 <u>+</u> 3	234 + 4	243 <u>+</u> 13
1.47	268 <u>+</u> 8	283 <u>+</u> 9	285 <u>+</u> 2	252 <u>+</u> 4	260 + 13
1.63	287 <u>+</u> 8	304 <u>+</u> 9	307 <u>+</u> 3	271 <u>+</u> 2	278 <u>+</u> 15
1.80	304 + 8	328 <u>+</u> 9	327 <u>+</u> 3	290 <u>+</u> 3	294 <u>+</u> 16
1.96	316 + 8	339 <u>+</u> 10	345 <u>+</u> 5	308 + 3	309 <u>+</u> 16
2.12	327 + 8	354 <u>+</u> 13	362 <u>+</u> 6	325 + 4	324 + 18
2.29	343 <u>+</u> 10	363 <u>+</u> 14	378 <u>+</u> 7	341 <u>+</u> 3	338 <u>+</u> 19
2.45	357 + 12	387 <u>+</u> 12	393 <u>+</u> 10	358 <u>+</u> 3	349 <u>+</u> 21
2.61	372 + 14	402 <u>+</u> 11	412 <u>+</u> 10	375 <u>+</u> 3	361 + 21

TABLE A5. Average Temperatures for Solar Furnace

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