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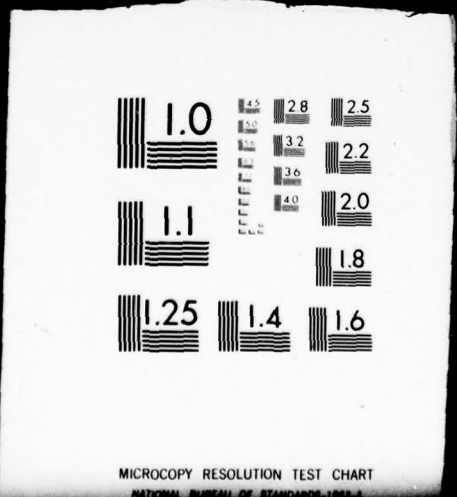
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DETERMINATION OF THEORETICAL SAMPLING EFFICIENCIES FOR
ASPIRATED PARTICULATE MATTER THROUGH A DRES
SAMPLING PROBE IN ANISOKINETIC FLOW (U)

by

Stanley B. Mellisen



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ABSTRACT

↓
Sampling efficiencies are calculated for an aspirated particulate matter sampling probe under various conditions of anisokinetic flow. A mathematical model developed for the purpose was used to obtain results for a wide range of particle sizes and flow velocities. The results can be used to predict or correct sampling errors in field or laboratory experiments. Using the same test parameters as in previous experimental tests by other workers, sampling efficiencies were calculated and the results were found to agree favorably with the results of the experiments.

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NOTATION

C	particle concentration in the sample, g cm^{-3}
C_0	particle concentration in the free stream, g cm^{-3}
d	particle diameter, cm
D	distance from the inlet to the outlet cross section of the collection tube, cm
h	thickness of the collection tube wall at the outlet cross section, cm
L	length of coaxial boundary tube, cm
r	radial co-ordinate of particle position, cm
r_A	radius of coaxial boundary tube, cm
r_B	radius of collection tube, cm
$r_{p,\infty}$	radial co-ordinate of particle position far upstream, cm
$r_{s,\infty}$	far upstream radius of the stream tube that impinges on the collection tube circumference, cm
t	time, seconds
u_r	radial component of local fluid velocity, cm sec^{-1}
u_z	axial component of local fluid velocity, cm sec^{-1}
U	fluid velocity in collection tube, cm sec^{-1}
U_A	fluid velocity at boundary tube entrance, cm sec^{-1}
U_B	fluid velocity at collection tube exit, cm sec^{-1}
U_C	fluid velocity at boundary tube exit, cm sec^{-1}
U_0	free stream velocity, cm sec^{-1}
v_r	radial component of local particle velocity, cm sec^{-1}
v_z	axial component of local particle velocity, cm sec^{-1}
z	axial co-ordinate (origin at collection tube inlet) of particle position, cm
z_0	axial co-ordinate of particle far upstream, cm

NOTATION (Cont'd)

μ	absolute viscosity of fluid, poise
ρ	fluid density, g cm^{-3}
σ	particle density, g cm^{-3}
ψ	stream function, $\text{cm}^3 \text{sec}^{-1}$

The following are dimensionless

C_D	drag coefficient for spheres
$G(1),$ $G(2),$ $G(3)$ and $G(4)$	dependent variables solved for by numerical integration. They represent $\bar{v}_z, \bar{v}_y, \bar{z}$ and \bar{r} respectively
E_m	collection efficiency of sampling tube
H	thickness of collection tube wall, h/r_A
i, j	grid point co-ordinates in the radial and axial directions respectively
i_B, j_B	grid point co-ordinates of the edge of the collection tube inlet
j_0	axial grid point co-ordinate of a particle at the far upstream position
K	inertia parameter of particle
\bar{r}	radial co-ordinate of particle, r/r_B
$\bar{r}_{p,\infty}$	radial co-ordinate of particle position far upstream, $r_{p,\infty}/r_b$
$\bar{r}_{s,\infty}$	far upstream radius of the stream tube that impinges on the collection tube circumference, $r_{s,\infty}/r_b$
R	radial co-ordinate used in calculating the stream function field, r/r_A
Re	spherical particle Reynolds number in flow in the proximity of the collection tube
Re_0	spherical particle Reynolds number in free stream

NOTATION (Cont'd)

\bar{u}_r	radial component of local fluid velocity, $d\bar{u}/d\bar{r}$
\bar{u}_z	axial component of local fluid velocity, $d\bar{u}/d\bar{z}$
\bar{v}_r	radial component of local particle velocity, $d\bar{r}/d\tau$
\bar{v}_z	axial component of local particle velocity, $d\bar{z}/d\tau$
\bar{z}	axial co-ordinate (origin at collection tube inlet) of particle, z/r_B
\bar{z}_0	axial co-ordinate of particle far upstream, z_0/r_B
Z	axial co-ordinate used in calculation of the stream function field, z/r_A
α	ratio of collection tube radius to boundary tube radius, r_B/r_A
β	length of coaxial boundary tube, L/r_A
γ	distance from the inlet to the outlet cross section of the collection tube, D/r_A
δ	distance from inlet of boundary tube to inlet of collection tube, $\beta - \gamma$
τ	time, tU_A/r_B
ϕ	dimensionless group independent of particle position, Re_0^2/K
ψ	stream function, $\psi/1/2U_A r_A^2$

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1. INTRODUCTION

The collection of a representative sample of finely divided particulate matter from still or moving airstreams is required where the size distribution, mass flow rate, concentration, or some other characteristic of the particulate-air system has to be determined. A sample of particulate matter will be representative only if the particle size distribution and content in the sample are the same as those in the ambient air at the point of sampling.

Particulate matter sampling instruments used at DRES consist essentially of a probe or tube through which the sample is drawn and then separated from the air stream for analysis.

Such a sampling system may be subject to three distinct types

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of error (Vitols, 1964) due to:

- (1) particles failing to enter the sampling probe in representative concentrations;
- (2) particles being deposited between the probe mouth and the separation location; and
- (3) particles being shattered, aggregated, or incompletely retained by collection devices.

Particles enter the sampling probe in representative concentrations when the entrance velocity is exactly equal to the velocity of the gas being sampled, in which case the sampling is said to be isokinetic. If the velocities are unequal errors of type (1) occur, in which case the sampling is referred to as anisokinetic.

The purpose of this report is to describe a mathematical model for calculating the error due to anisokineticity for a sampling probe developed and used at DRES (Fig. 1). The nominal inside diameter of this probe is 3/4 inch. The inlet end is sharp edged and the outside surface is sloped so that the tube wall thickness increases away from the inlet with a 1 in 12 slope. The length of the tapered section is 2 3/4 inches. Results are provided for various flow conditions so that the user of the sampling probe can determine the magnitude of sampling errors and make corrections when anisokinetic sampling occurs.

2. DEFINITION OF THE PROBLEM

Due to inertial and drag forces, a particle flowing at the free stream velocity far upstream of the sampling probe will not necessarily follow a stream line in the vicinity of the probe, where radial and axial velocity components of the fluid may be changing markedly. Thus, if the estimated concentration of particles in the free stream is taken as the number of particles collected by the sampler divided by the volume of air passing through the probe, the calculated values may differ markedly from the true free stream values. The total volume of air passing through the probe will be that enclosed by a stream tube that impinges on the outer circumference of the probe opening. When the velocity at the tube inlet,

U_B is less than the free stream velocity, U_A ($U_B/U_A < 1$), particles from outside the limiting stream tube will enter the sampler, while for $U_B/U_A > 1$ particles originally inside the limiting stream tube will pass outside the probe. All particles of the same diameter that are collected by the probe are those within the circular envelope generated by particles that just impinge on the outer circumference of the probe. Let $r_{p,\infty}$ be the upstream radius of the limiting particle trajectory envelope, and $r_{s,\infty}$ be the upstream radius of the stream tube that impinges on the probe circumference. Then the sampling efficiency is given by:

$$\left(\frac{C}{C_0}\right) = \left(\frac{r_{p,\infty}}{r_{s,\infty}}\right)^2 \quad (\text{Eq. 1})$$

where C_0 is the particle concentration in the free stream and C is the particle concentration in the sample.

The collection efficiency of the probe is given by:

$$E_m = \left(\frac{r_{p,\infty}}{r_B}\right)^2 \quad (\text{Eq. 2})$$

where r_B is the radius of the probe inlet. The problem then is to calculate $r_{p,\infty}$ and $r_{s,\infty}$ so that the sampling efficiency can be found from Eq. 1 and, incidentally, so that the collection efficiency can be found from Eq. 2.

3. EQUATIONS OF MOTION

The motion of an individual particle has been shown (Vitols, 1964 and Batchelor, 1956) to be determined by the following ordinary differential equations:

$$\frac{d\bar{v}_r}{d\tau} = \frac{C_D \text{Re}(\bar{u}_r - \bar{v}_r)}{24 K} \quad (\text{Eq. 3})$$

$$\frac{d\bar{v}_z}{d\tau} = \frac{C_D \text{Re}(\bar{u}_z - \bar{v}_z)}{24 K} \quad (\text{Eq. 4})$$

where $Re = Re_0 [(\bar{u}_r - \bar{v}_r)^2 + (\bar{u}_z - \bar{v}_z)^2]^{1/2}$ (Eq. 5)

$$K = \frac{\sigma d^2 U_A}{18 \mu r_B} \quad \text{particle inertia parameter} \quad (\text{Eq. 6})$$

$$Re_0 = \frac{U_A d \rho}{\mu} \quad \text{free stream Reynolds number} \quad (\text{Eq. 7})$$

The symbols are defined in the notation section near the front of this report and the basic geometry of the flow system is illustrated in Fig. 2.

Several assumptions are inherent in the use of Eqs. 3 and 4 for calculating the collection and sampling efficiencies due to a stream of particles, including:

- (a) uniform particle distribution;
- (b) no gravitational or electrostatic forces of consequence;
- (c) monodisperse spherical particles with diameter very small in relation to the inlet diameter of the probe; and
- (d) free stream flow that is steady, incompressible and irrotational.

The drag coefficient is a function of Reynolds number and is available in the form of definitive empirical equations (Davies, 1945). These equations are stated as follows:

$$Re = \frac{C_D Re^2}{24} - 2.3363 \times 10^{-4} (C_D Re^2)^2 + 2.0154 \times 10^{-6} (C_D Re^2)^3 - 6.9105 \times 10^{-9} (C_D Re^2)^4 \quad (\text{Eq. 8})$$

for $Re < 4$ or $C_D Re^2 < 140$

$$\log_{10} Re = -1.29536 + 9.86 \times 10^{-1} (\log_{10} C_D Re^2) - 4.6677 \times 10^{-2} (\log_{10} C_D Re^2)^2 + 1.1235 \times 10^{-3} (\log_{10} C_D Re^2)^3 \quad (\text{Eq. 9})$$

for $3 < Re < 10^4$ or $C_D Re^2 < 4.5 \times 10^7$

4. AIR FLOW FIELD EQUATIONS

The equations of fluid velocity were derived from the stream function for ideal flow over and through the collection probe. To solve the problem, an outer boundary was used around the collection tube in the form of a coaxial tube of radius r_A (Fig. 3), which was chosen large enough so that the effect of the boundary tube on flow in the proximity of the collection tube is negligible. The collection tube was inserted a distance D into the downstream end of the boundary tube. Since the flow is axisymmetric only a radial plane containing both tubes has to be considered.

The fluid enters the boundary tube with steady velocity U_A , and separates into a central stream with velocity U_B at the entrance of the collection tube, and an annular stream, with velocity U_C , at the downstream end of the boundary tube. The axial velocities U_A , U_B and U_C are uniform. Also, there is no radial flow at the end cross sections.

The boundary conditions on the flow can now be completely specified so that the flow field can be obtained by solution of the equation of the stream function.

The axially symmetric stream function $\psi(r,z)$ (Batchelor, 1967) satisfies:

$$\frac{\partial^2 \psi}{\partial r^2} - \frac{1}{r} \frac{\partial \psi}{\partial r} + \frac{\partial^2 \psi}{\partial z^2} = 0 \quad (\text{Eq. 10})$$

The two velocity components (Fig. 2) are given by:

$$u_z = \frac{1}{r} \frac{\partial \psi}{\partial r} \quad (\text{Eq. 11})$$

$$u_r = - \frac{1}{r} \frac{\partial \psi}{\partial z} \quad (\text{Eq. 12})$$

When U_A and U_B are specified continuity gives U_C as follows:

$$U_C = \frac{U_A - \alpha^2 U_B}{1 - \left[\left(\frac{r_B + h}{r_B} \right) \alpha \right]^2} \quad (\text{Eq. 13})$$

$$\text{where } \alpha = \frac{r_B}{r_A} \quad (\text{Eq. 14})$$

and h is the thickness of the collection tube wall at the outlet end (Fig. 3).

For uniform velocity profiles, the stream function is of the form:

$$\psi = \frac{1}{2} u r^2 \quad (\text{Eq. 15})$$

To allow for greater generality, the stream function and the geometric variables were restated in the following dimensionless form:

$$\Psi = \frac{\psi}{\frac{1}{2} U_A r_A^2} \quad (\text{Eq. 16})$$

$$R = \frac{r}{r_A} \quad (\text{Eq. 17})$$

$$Z = \frac{z}{r_A} \quad (\text{Eq. 18})$$

$$\beta = \frac{L}{r_A} \quad (\text{Eq. 19})$$

$$\gamma = \frac{D}{r_A} \quad (\text{Eq. 20})$$

$$H = \frac{h}{r_A} \quad (\text{Eq. 21})$$

The boundary values for the stream function and the geometric configuration in terms of the dimensionless variables are shown in Fig. 4.

The axially symmetric stream function equation (Eq. 10) becomes:

$$\frac{\partial^2 \Psi}{\partial R^2} - \frac{1}{R} \frac{\partial \Psi}{\partial R} + \frac{\partial^2 \Psi}{\partial Z^2} = 0 \quad (\text{Eq. 22})$$

5. DISCRETIZATION SCHEME FOR THE AIR FLOW FIELD

The equation for the axially symmetric stream function (Eq. 22) is discretized as follows:

$$\frac{\psi_{i-1,j} - 2\psi_{i,j} + \psi_{i+1,j}}{\Delta R^2} - \frac{\psi_{i+1,j} - \psi_{i-1,j}}{2i\Delta R^2} + \frac{\psi_{i,j-1} - 2\psi_{i,j} + \psi_{i,j+1}}{\Delta Z^2} = 0 \quad (\text{Eq. 23})$$

where i and j are the grid point numbers in the R and Z directions respectively (Fig. 5). Eq. 23 can be rearranged to give a simple equation by choosing a square grid so that ΔZ and ΔR are equal. The resulting equation, which is suitable for Gauss-Seidel iteration (Carnahan et al., 1969), is given as follows:

$$\psi_{i,j} = \frac{\psi_{i-1,j} + \psi_{i+1,j} + \psi_{i,j-1} + \psi_{i,j+1}}{4} - \frac{\psi_{i+1,j} - \psi_{i-1,j}}{8i} \quad (\text{Eq. 24})$$

Eq. 24 is suitable for all interior points, which are defined as points for which the nearest boundary is at least one grid size unit away. A grid can be chosen such that all points not right on the boundary are interior points, with the exception of the outside boundary of the collection tube which has a slope of $1/12$. To handle these points, which herein are called special boundary points (Fig. 6), a Taylor series expansion for a point near the boundary was used, as is generally applied to curved boundaries (Carnahan et al., 1969). The resulting finite difference equation is suitable for these points instead of Eq. 24.

$$\psi_{i,j} = \frac{a}{2(a+1)} \left[\psi_{i,j-1} + \psi_{i,j+1} + \frac{2\psi_A}{a(a+1)} + \frac{2\psi_{i+1,j}}{a+1} + \frac{\psi_A - \psi_{i+1,j}}{i(1+a)} \right] \quad (\text{Eq. 25})$$

where a is the distance, in terms of the grid size unit, from the grid point to the sloping boundary at point A and $0 < a < 1$.

The grid size was chosen so that the thickness of the collection tube wall at the downstream end corresponded to exactly two units. Then the length of the tube must be 24 grid units since the slope of the outer collection tube boundary is $1/12$. The boundary tube radius

and the distance, δ , (Fig. 4) to the upstream end of the boundary tube were chosen so that they are at least five times the radius of the inlet to the collection tube, so that particles travelling in the flow toward the tube inlet will behave as if they are coming from far upstream and into the collection tube in free space. Now, to satisfy these conditions, and to ensure adequate core storage space in the DRES IBM 1130 computer, while maintaining a fineness of grid for sufficient accuracy, the collection tube radius was chosen to be 12 units, the boundary tube radius 60 units, and the boundary tube length 88 units (Fig. 5).

The stream function at each point was then obtained by means of Gauss-Seidel iteration (Carnahan et al., 1969) using Eq. 24 and Eq. 25. The boundary conditions were set initially and held fixed throughout the course of the solution (Fig. 4) and, as a starting point for the iteration, the value of the stream function at all points not right on the boundary was set to zero. The calculations were done with the DRES IBM 1130 computer by means of a Fortran program, the listing of which is shown in Appendix A.

6. SOLUTION OF THE EQUATIONS OF MOTION

As previously stated in Section 2 of this report, the problem is to find the upstream radii $r_{p,\infty}$ and $r_{s,\infty}$ so that the sampling and collection efficiencies can be calculated. In the same dimensionless form used in Eqs. 3 and 4 the value of $\bar{r}_{p,\infty}$ (notation) was found by an iterative procedure called the half interval method (Carnahan et al., 1969). The value of $\bar{r}_{p,\infty}$ for a critical particle was estimated far upstream, the path followed to the plane of the collection tube opening, and the miss distance calculated. Next, the half interval method, previously mentioned, was applied to determine a better initial estimate. Then the path was followed again to the plane of the collection tube opening for another calculation of the miss distance. This process was repeated several times until sufficient accuracy was achieved. The initial upstream position in a plane perpendicular to the flow direction was located far enough from the collection tube opening so that free stream conditions prevailed. A distance of five target radii upstream of the target centre was considered

adequate (Batchelor, 1956).

The path of an individual particle was determined step-by-step by applying a fourth order Runge-Kutta method (Carnahan et al., 1969) to the equations of motion (Eqs. 3 and 4). The values of Re and K in these equations were easily found for each new step by direct substitution of previously determined values into Eqs. 5, 6 and 7, but the value of $C_D Re$ in Eqs. 3 and 4 had to be calculated in each step by numerical solution of the definitive empirical equations (Eqs. 8 and 9). This was done using Newton's method (Carnahan et al., 1969) for finding the zero of a function. The values of \bar{u}_r and \bar{u}_z were calculated in each step from the stream function field as follows:

$$\bar{u}_r = \frac{\Psi_{i,j-1} - \Psi_{i,j+1}}{4(i-1)(\Delta R)^2} \quad (\text{Eq. 26})$$

$$\bar{u}_z = \frac{\Psi_{i+1,j} - \Psi_{i-1,j}}{4(i-1)(\Delta R)^2} \quad (\text{Eq. 27})$$

where i and j define the grid point of the particle position. Since the radius of the collection tube was chosen to be 12 grid units these are given by:

$$i = 1 + 12\bar{r} \quad (\text{Eq. 28})$$

$$j = j_0 + 12(\bar{z} - \bar{z}_0) \quad (\text{Eq. 29})$$

where j_0 and \bar{z}_0 are the starting point values of j and \bar{z} . The values of i and j obtained from Eqs. 28 and 29 were rounded off to the nearest lower integer value in each calculation. The value of $\bar{r}_{s,\infty}$ was obtained directly from the stream function by:

$$\bar{r}_{s,\infty} = \frac{\Delta R(i-2)r_A}{r_B} \sqrt{\frac{\Psi_{i_B,j_B}}{\Psi_{i-1,j_0}}} \quad (\text{Eq. 30})$$

calculated at the lowest value of i satisfying:

$$\Psi_{i,j_0} > \Psi_{i_B,j_B} \quad (\text{Eq. 31})$$

where i_B and j_B define the grid point at the edge of the collection tube inlet. The calculations to obtain the solutions were done with the DRES IBM 1130 computer by means of a Fortran program, the listing of which is shown in Appendix B. The sampling and collection efficiencies given by Eqs. 1 and 2 were also obtained by this program after the values of $\bar{r}_{p,\infty}$ and $\bar{r}_{s,\infty}$ had been calculated.

7. RESULTS

A sample stream function field for one set of input data is shown after its associated computer program (Appendix A), and a sample calculation of the sampling and collection efficiencies is shown after their associated computer program (Appendix B). Using these two computer programs many more calculations were made to produce the graphical results shown in Figs. 7 to 14. These results are described in greater detail as follows.

Many experimenters have measured sampling efficiencies over the last sixty-five years (Vitols, 1964). An experimental study using zinc sphere test dust in a wind tunnel was done for sampling tubes of 0.65 to 1.90 cm diameter (Badzioch, 1959). The tubes were blunt-edged with a wall thickness of 0.6 mm. The results for some of these experiments are shown in Fig. 7. Using the same test parameters as in these experiments, sampling efficiencies were calculated for various velocity ratios, U_B/U_A , and the results plotted in Fig. 7 assuming that U_B/U_A , of the mathematical model, is equivalent to U/U_0 in the experiments of Badzioch. The results agree favorably even though the tube shapes were slightly different. However, calculations for various input parameters, keeping $U_B/U_A = 1$, gave sampling efficiencies between 0.9900 and 1.0000, indicating that the tapered outside wall of the collection tube does not disturb the flow so as to cause deviations for isokinetic velocity conditions. Therefore, the two tubes can, for practical purposes, be considered equivalent.

The sampling and collection efficiencies are functions of two dimensionless groups, the inertial parameter, K , and the free stream particle Reynolds number, Re_0 . A new dimensionless group:

$$\phi = \frac{Re_o^2}{K} \quad (\text{Eq. 32})$$

independent of particle size can be introduced (Friedlander, 1977). According to the rules of dimensional analysis this is permissible, but the efficiencies are still determined by two groups chosen to be K and ϕ . The calculated sampling efficiency is plotted against the inertia parameter in Figs. 8 and 9 for sampling rates of 5 and 40 litres per minute respectively. Curves are plotted for values of U_B/U_A of 1/3, 1, 3 and 10 in each of the two figures. Note that equal values of U_B/U_A correspond to a different value of ϕ in Figs. 8 and 9. This is due to two different sampling rates because sampling rate and the corresponding free stream velocity were the only dimensional parameters which were changed. The calculated collection efficiencies were plotted in Figs. 10 and 11 for the same velocity ratios and sampling rates as the sampling efficiencies shown in Figs. 8 and 9.

To show the effect of varying the flow velocity in the sampling tube while maintaining a constant velocity ratio, the sampling efficiency was plotted against the inertia parameter in Fig. 12. Curves were plotted for the two sampling rates of 5 and 40 litres per minute as before. Similar curves are shown for the collection efficiencies in Fig. 13.

To illustrate directly the drastic effect of varying free stream velocity on the sampling efficiency for constant sampling velocity and particle size the sampling efficiency was plotted against the velocity ratio, U_B/U_A , for a particle size of 100 μm in Fig. 14.

8. DISCUSSION

As can be seen in Fig. 7, where the experiments of Badzioch are compared to the results obtained by means of the present mathematical model, the effect of anisokineticity produces sampling errors of greater than 100 percent even when the sampling velocity differs from the free stream velocity by less than a factor of three. These results are for 23 micron spherical particles. Smaller particles give rise to smaller

sampling error. The exact errors for very small particles cannot be determined by the present model because the computing errors increase with decreasing particle size for very small particles. Experience shows that, in addition to the requirement of a small grid size for calculating the stream function field, reliable results can only be achieved if the values of the time step increment $d\tau$ is at least less than the inertia parameter K . So, as particle size decreases, and correspondingly K decreases, a larger number of calculations are required which, in itself, can give rise to computing errors. However, the results did indicate that the sampling errors for particles in the order of 10 microns can be at least 50 percent. For example, for a sampling rate of 5 litres per minute and a sampling velocity which is a third of the free stream velocity, Fig. 8 shows a value of the sample concentration which is 1.5 times the free stream concentration when the inertia parameter is 0.029. This corresponds to a particle size of 10 microns when the particle density is 1 g cm^{-3} and 5 microns when the particle density is 4 g cm^{-3} . The density of zinc sulfide, which is sometimes used as phosphorescent trace material with particle sizes of about 5 microns, is 4.1 g cm^{-3} (Sehmel, 1973).

Although the model cannot be used for calculations using completely still air, very low free stream velocities can be treated. For a sampling rate of 5 litres per minute and a velocity ratio $U_B/U_A = 10$, as used in one of the curves in Fig. 8, the corresponding free stream velocity is 3 cm sec^{-1} . This is about the wind velocity expected in an enclosure such as an ordinary, normally ventilated, room.

Another point to note, when considering sampling errors, is that even when the velocity ratio and particle size are kept constant, the sampling error varies somewhat with the absolute value of the sampling tube velocity. This is indicated by the difference in the two curves (Fig. 12) for sampling rates of 5 and 40 litres per minute.

The drastic effect on sampling efficiency of varying the free stream velocity while maintaining a constant sampling velocity is illustrated in Fig. 14 where the calculated results for a particle size of 100 microns are plotted directly.

9. CONCLUSIONS

The effect of anisokineticity on sampling with the DRES sampling probe used in field conditions is large enough so that drastic errors in sampling can occur. The results obtained by the mathematical model described herein can be used to estimate the size of these sampling errors. They can also be used to correct measured samples if the wind velocity and sampling rate are also measured, and the particle size and density are known. Long sampling periods are best, so that effects due to wind speed changes which occur over short time periods are averaged out.

10. REFERENCES

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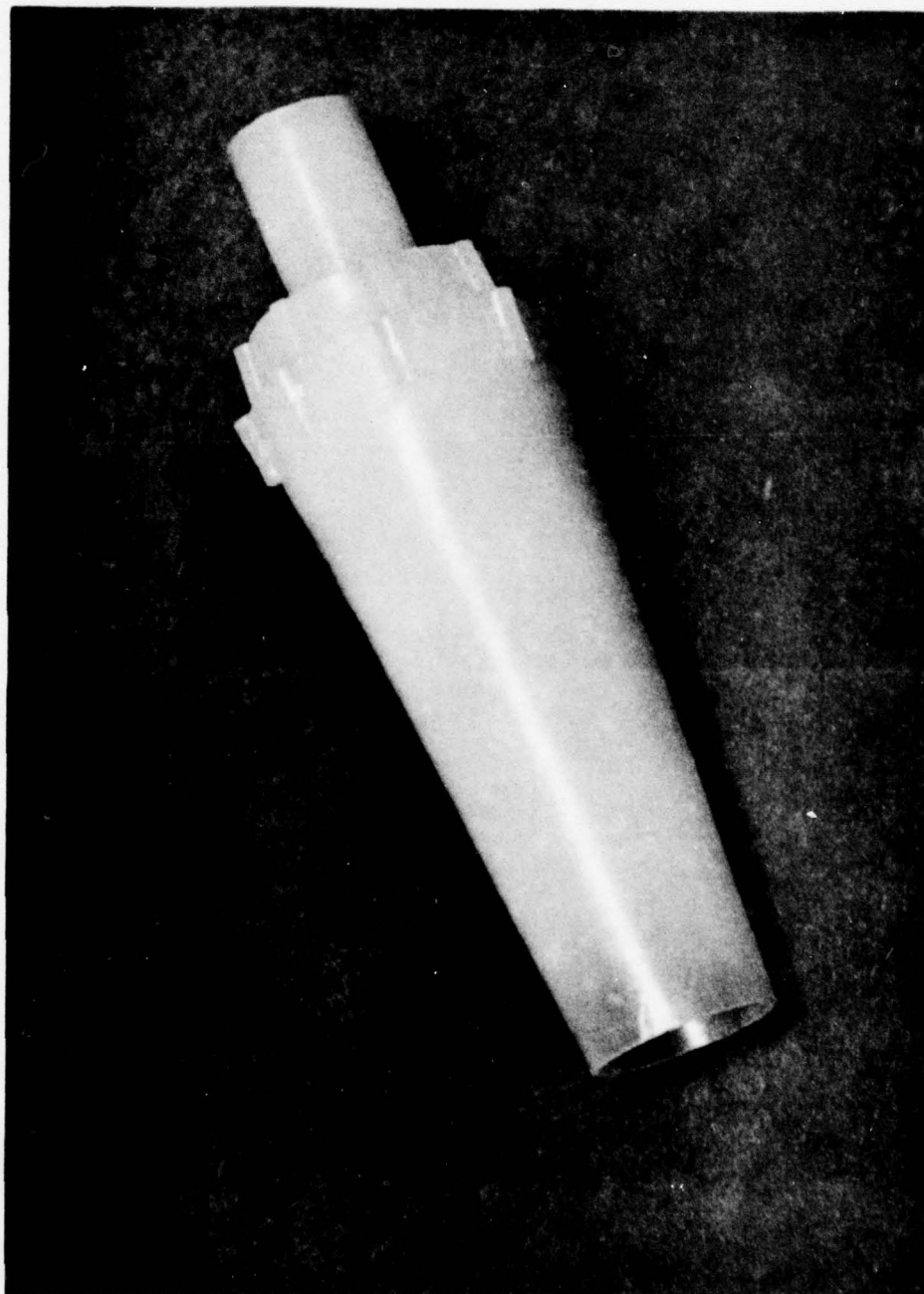


Figure 1: Collection Tube for Sampling Particulates in Air

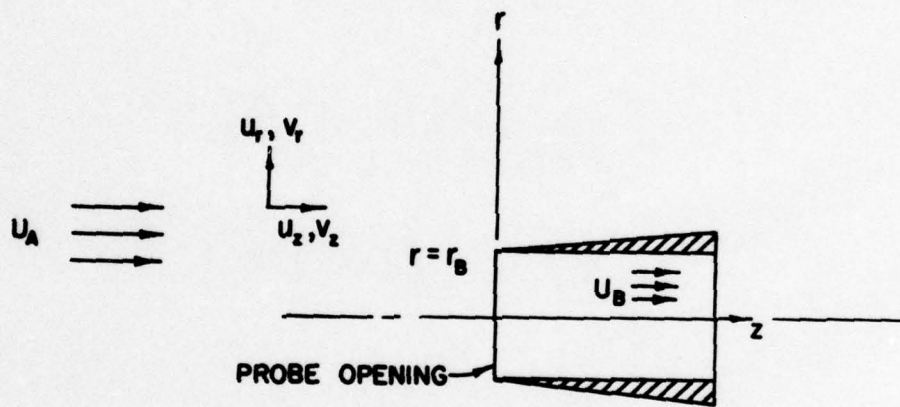


FIGURE 2: CO-ORDINATE SYSTEM FOR AXIAL FLOW
IN THE PROXIMITY OF THE COLLECTION TUBE

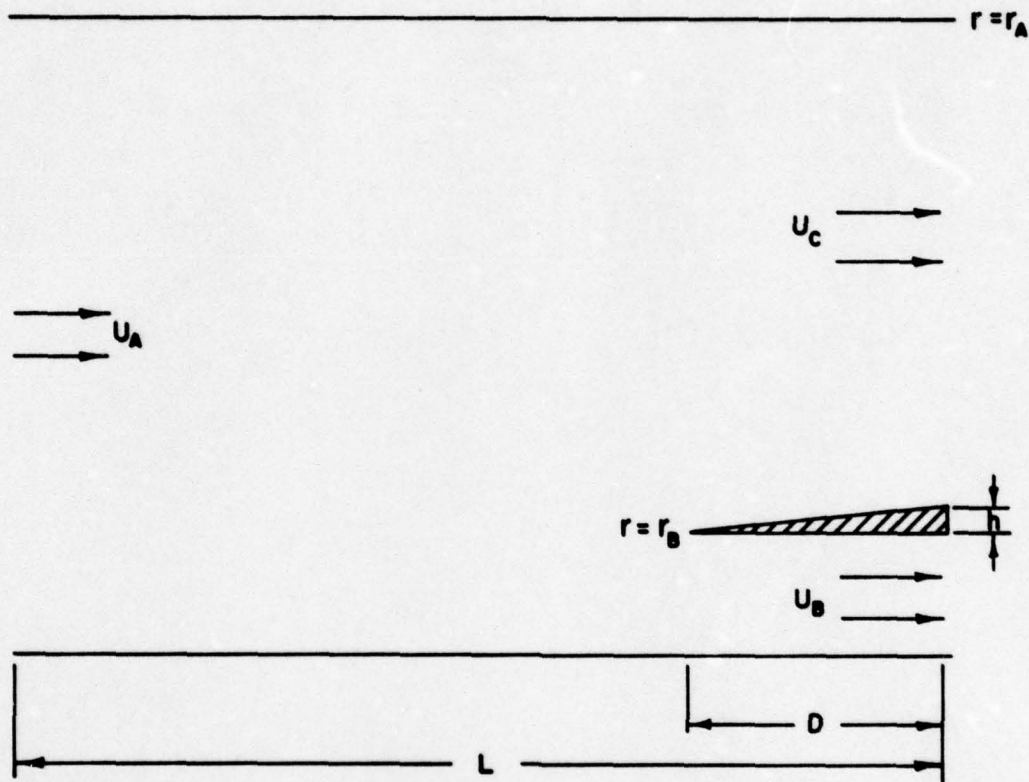


FIGURE 3: GEOMETRIC MODEL FOR FLOW FIELD IN THE PROXIMITY OF THE COLLECTION TUBE

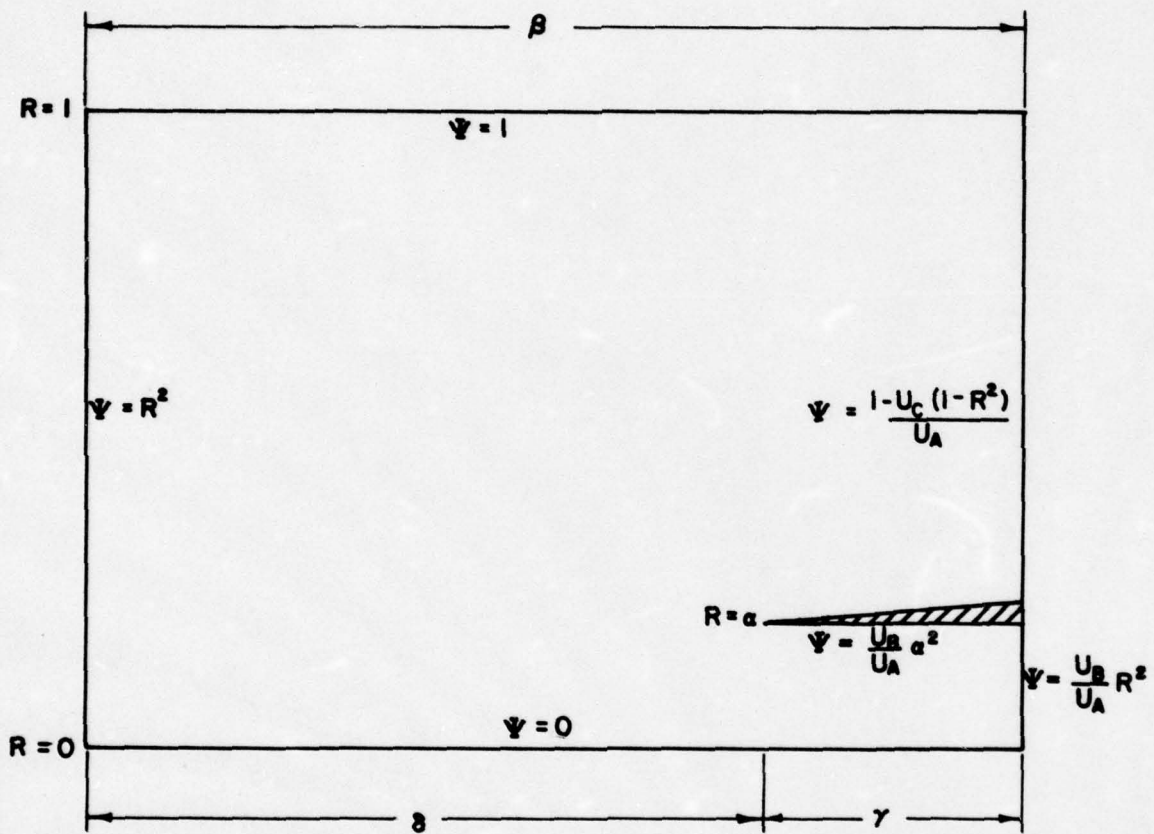


FIGURE 4: STREAM FUNCTION BOUNDARY CONDITIONS

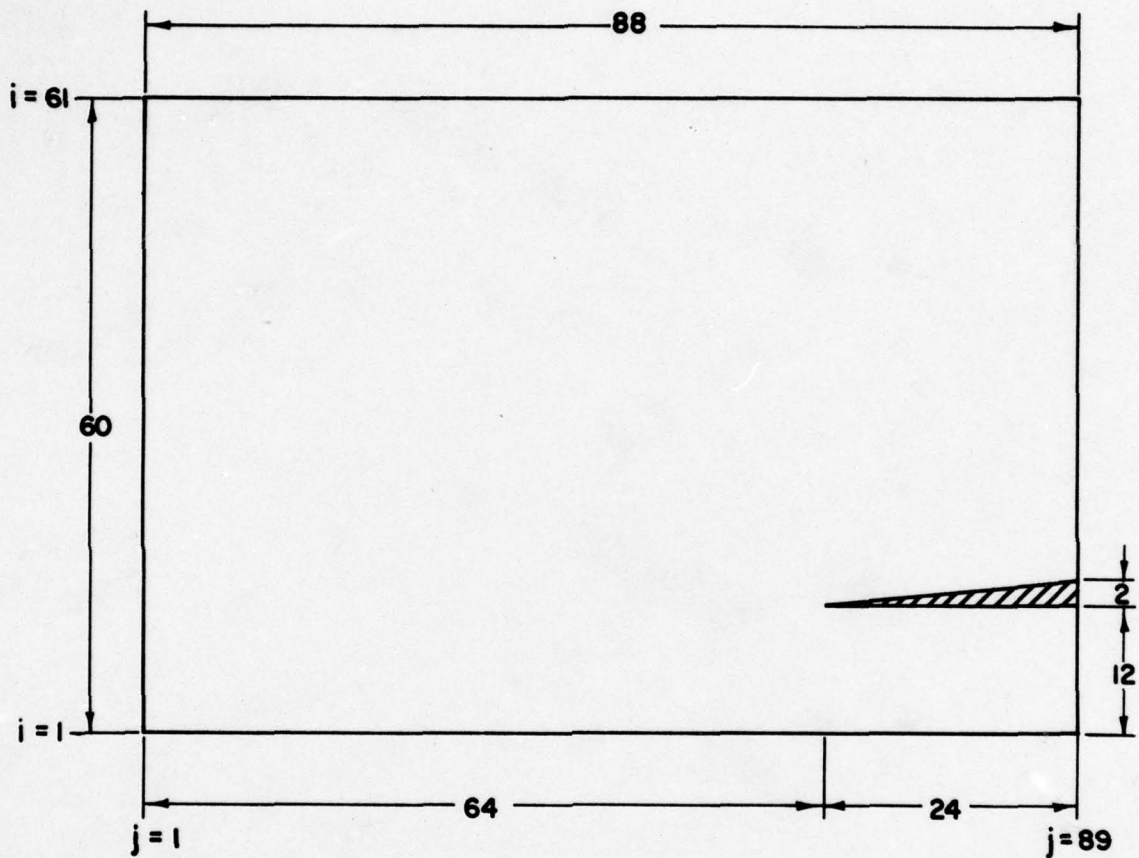
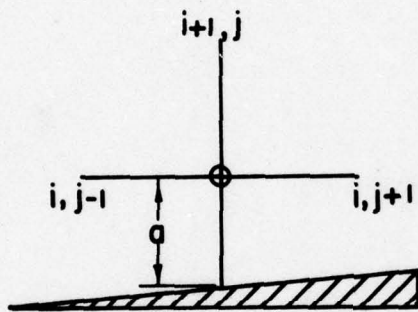


FIGURE 5: DIMENSIONS OF DISCRETIZATION GRID FOR AIR FLOW FIELD



O - SPECIAL BOUNDARY POINTS

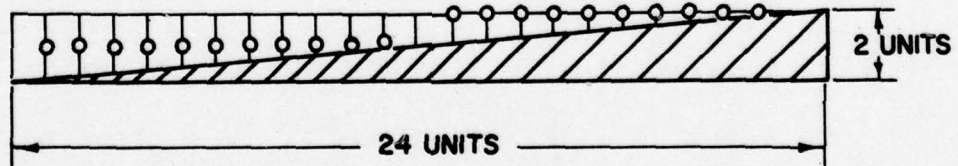


FIGURE 6: GRID POINTS ALONG SLOPING BOUNDARY OF COLLECTION TUBE

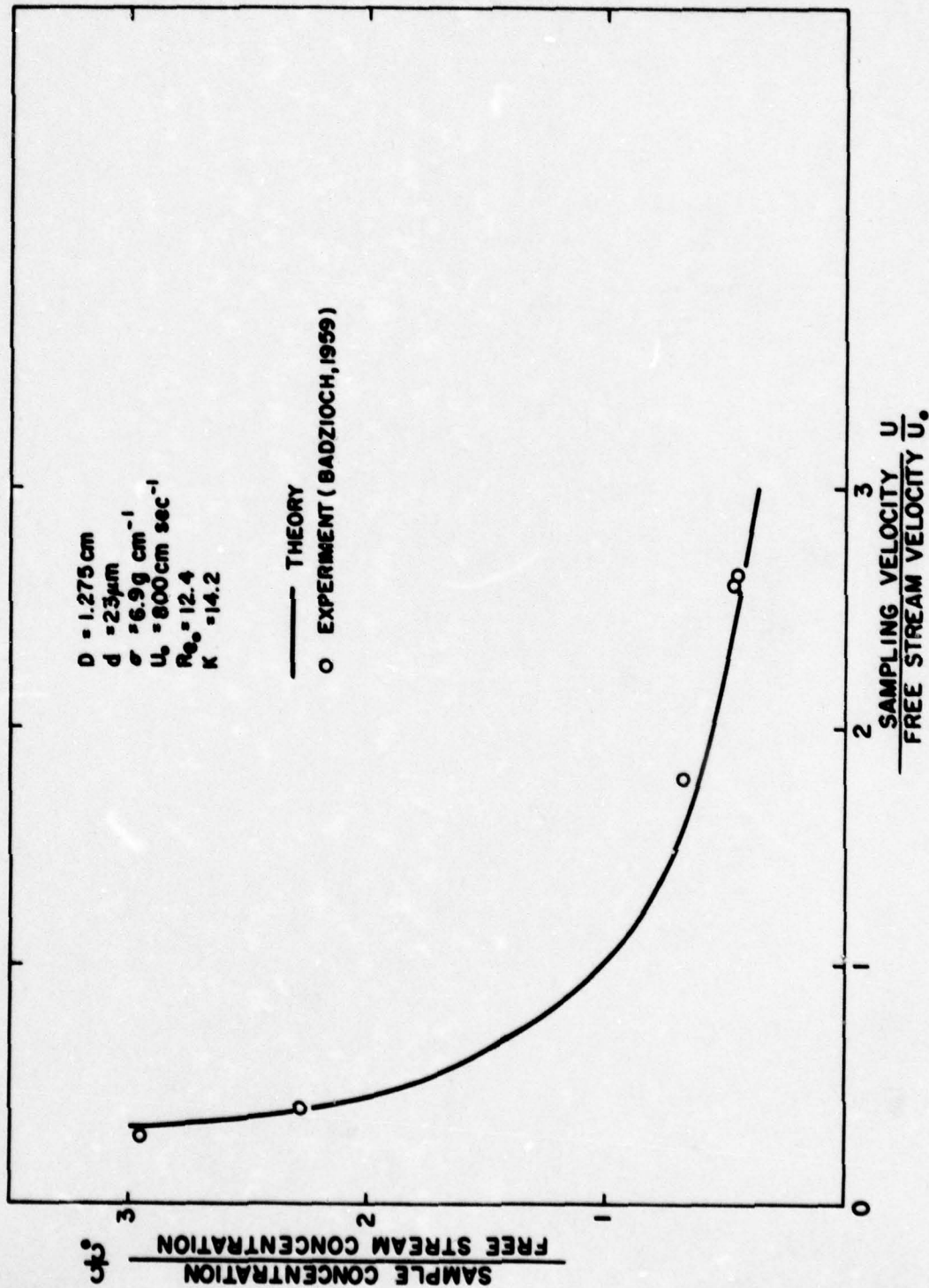


FIGURE 7: COMPARISON OF THEORETICAL AND EXPERIMENTAL EFFECTS OF ANISOKINETIC SAMPLING FOR ZINC SPHERE TEST DUST

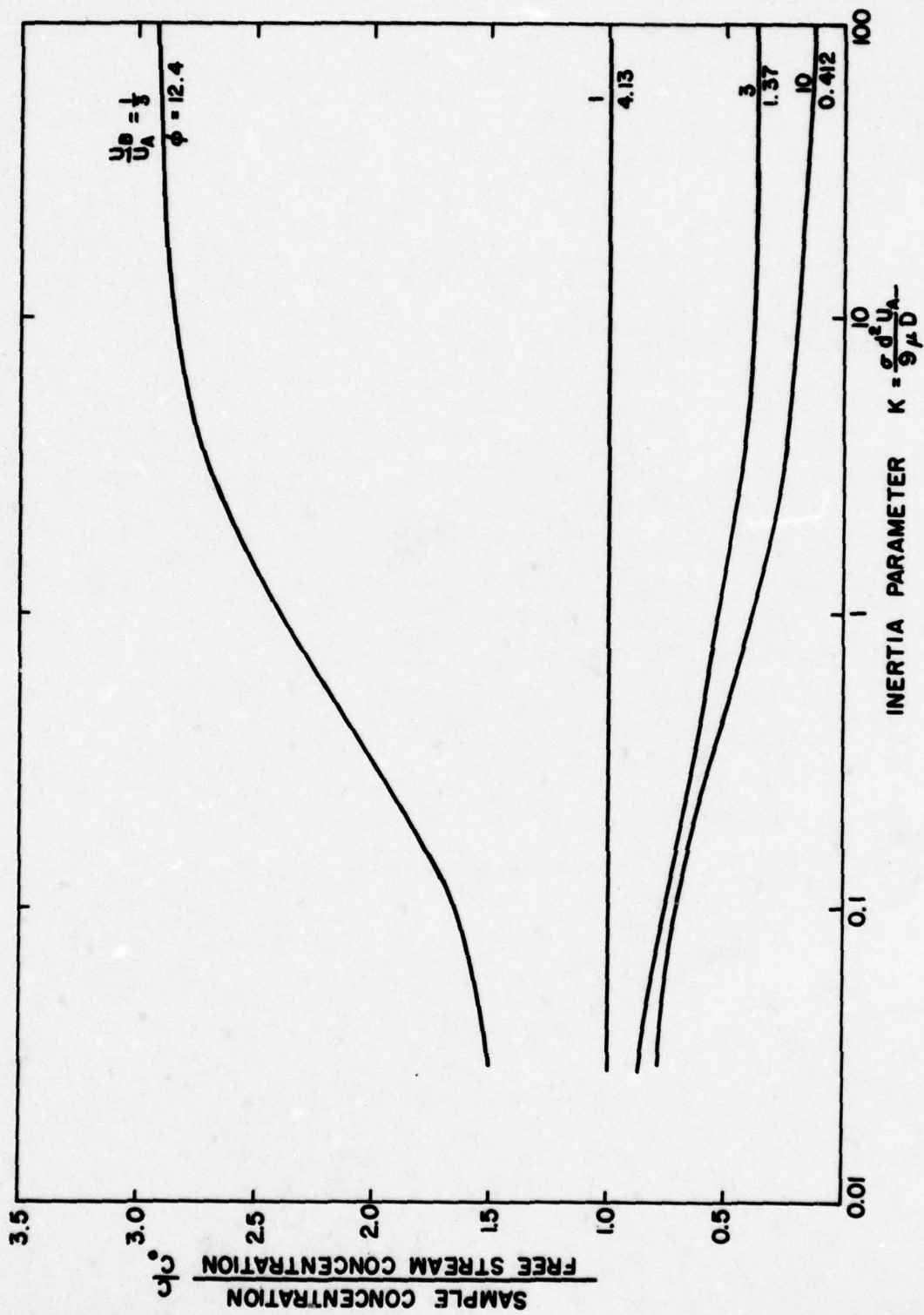


FIGURE 8: EFFECT OF VELOCITY RATIO ON SAMPLING EFFICIENCY FOR FIXED SAMPLING RATE (5 L min⁻¹)

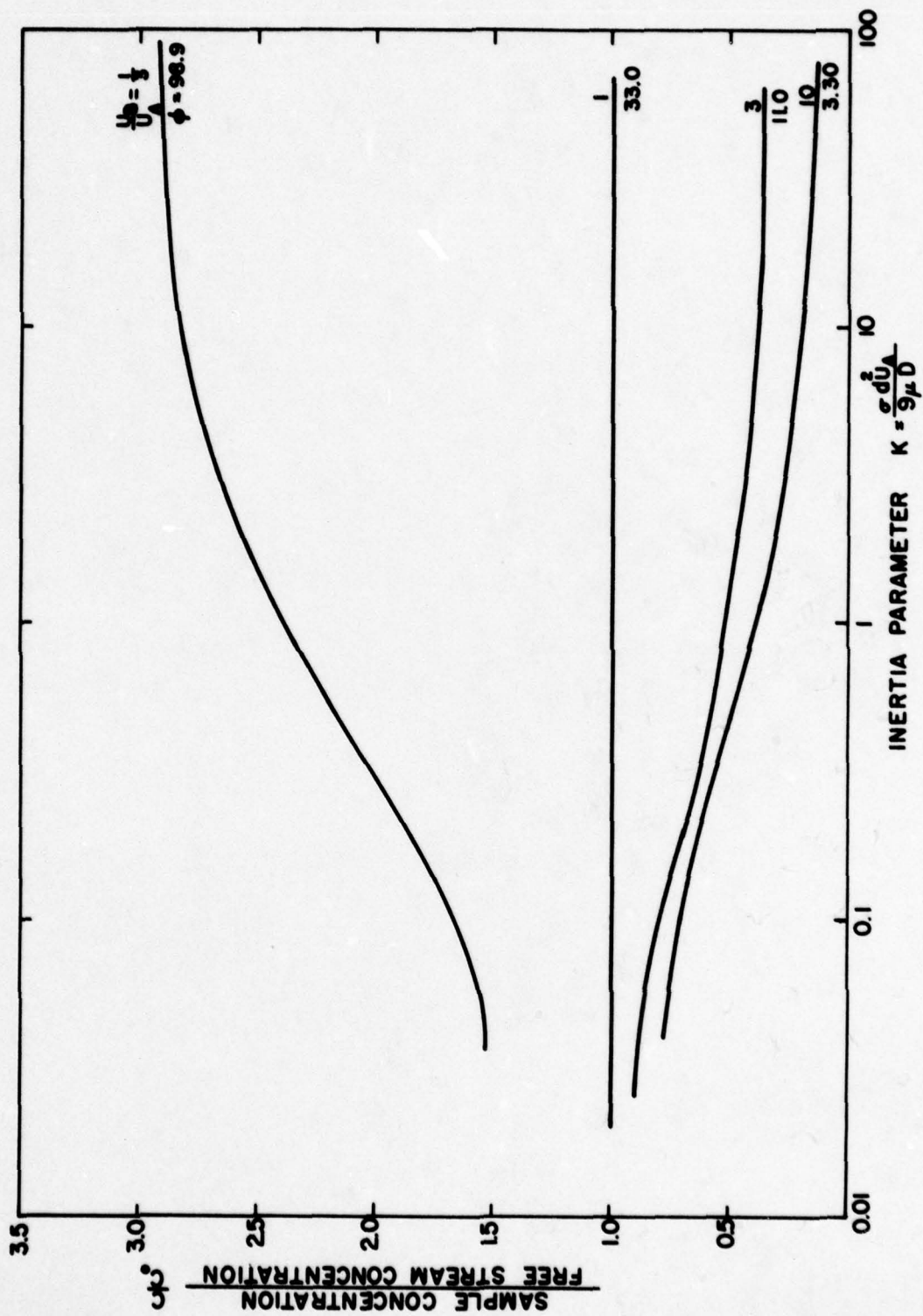


FIGURE 9: EFFECT OF VELOCITY RATIO ON SAMPLING EFFICIENCY FOR FIXED SAMPLING RATE (40 L min⁻¹)

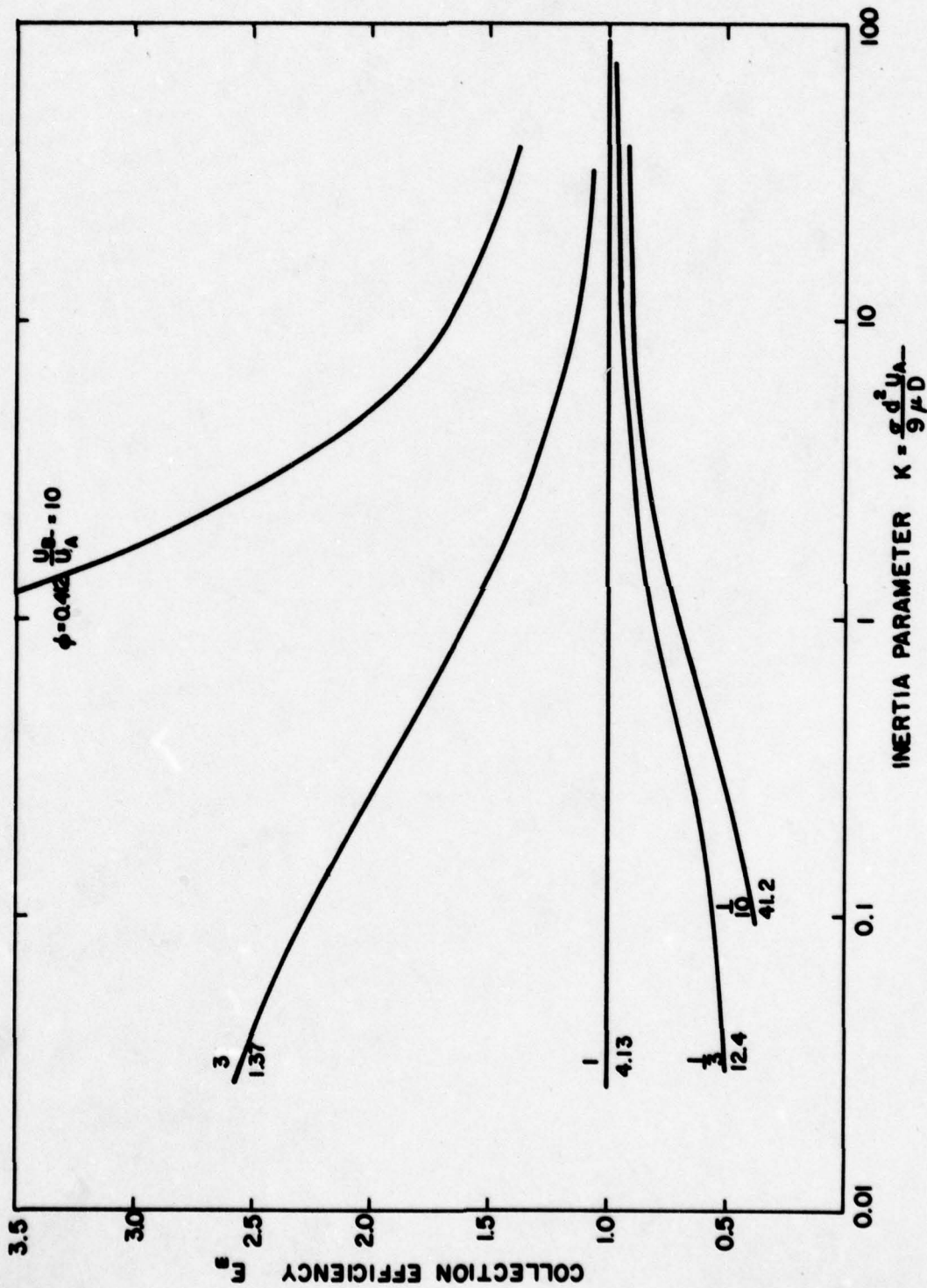


FIGURE 10: EFFECT OF VELOCITY RATIO ON COLLECTION EFFICIENCY FOR FIXED SAMPLING RATE (5L min⁻¹)

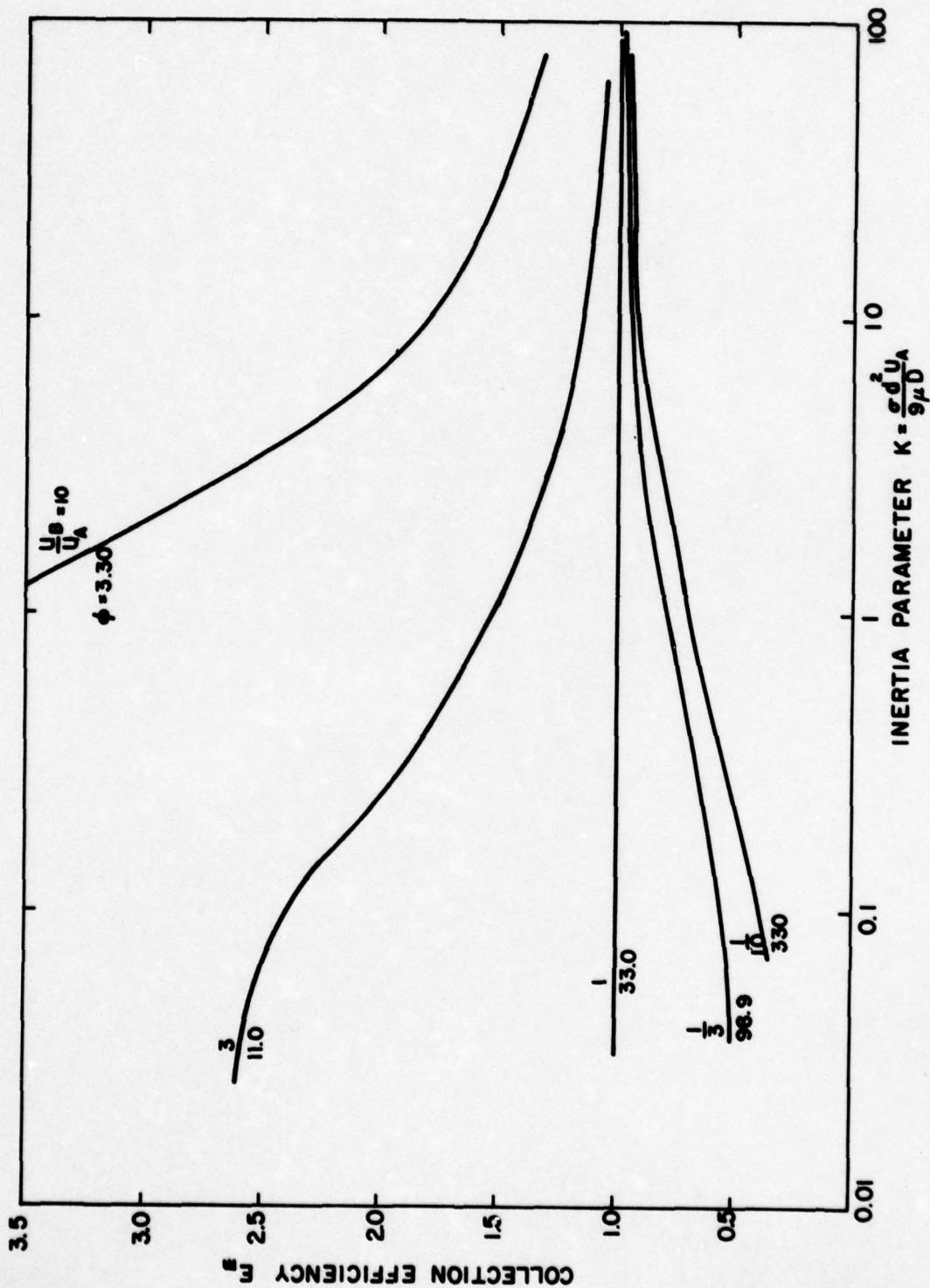


FIGURE II: EFFECT OF VELOCITY RATIO ON COLLECTION EFFICIENCY FOR FIXED SAMPLING RATE (40 L min⁻¹)

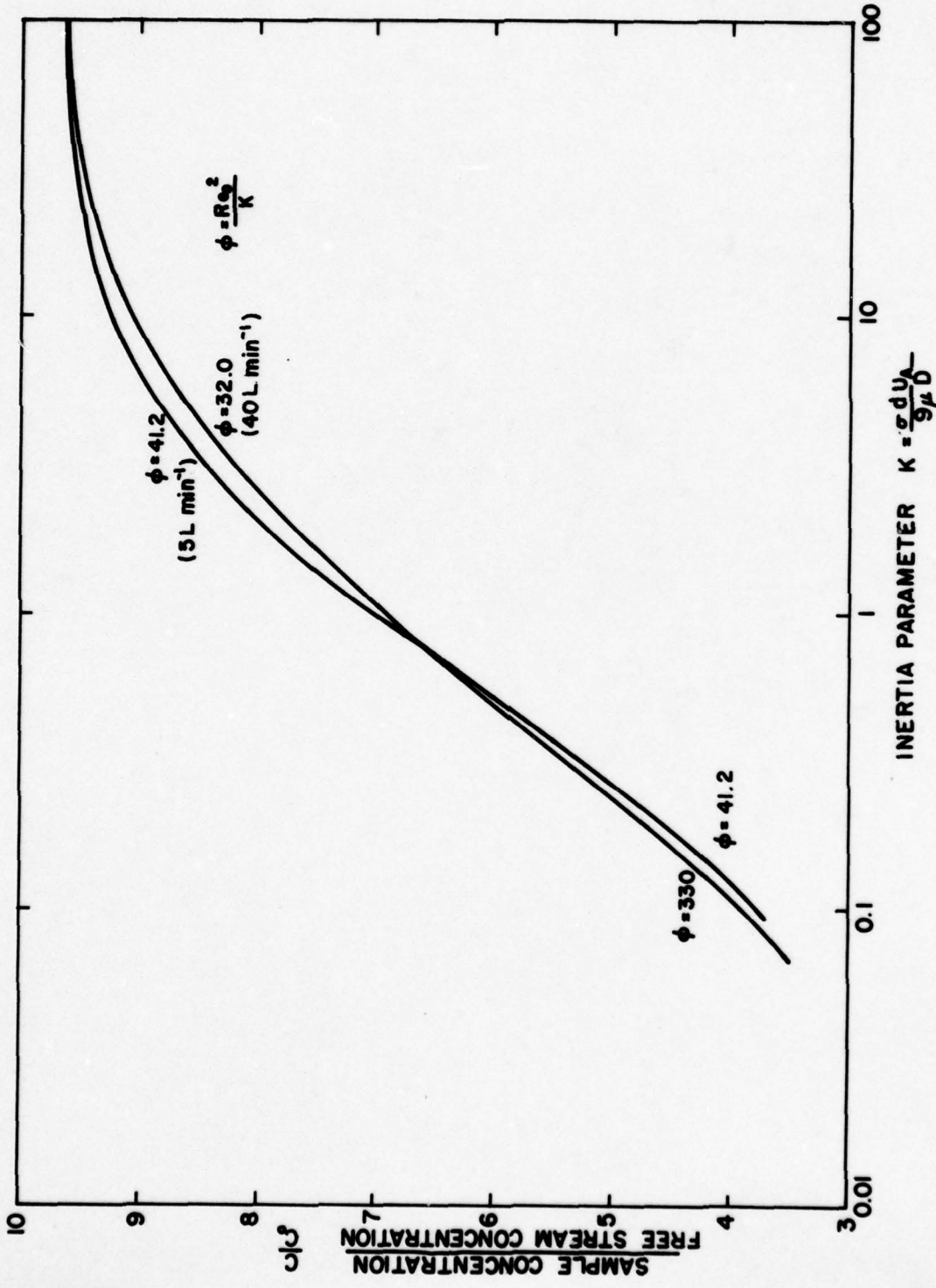


FIGURE 12: EFFECT OF FLOW VELOCITY IN SAMPLING TUBE ON SAMPLING EFFICIENCY FOR FIXED VELOCITY RATIO ($\frac{U_a}{U_\infty} = \frac{1}{10}$)

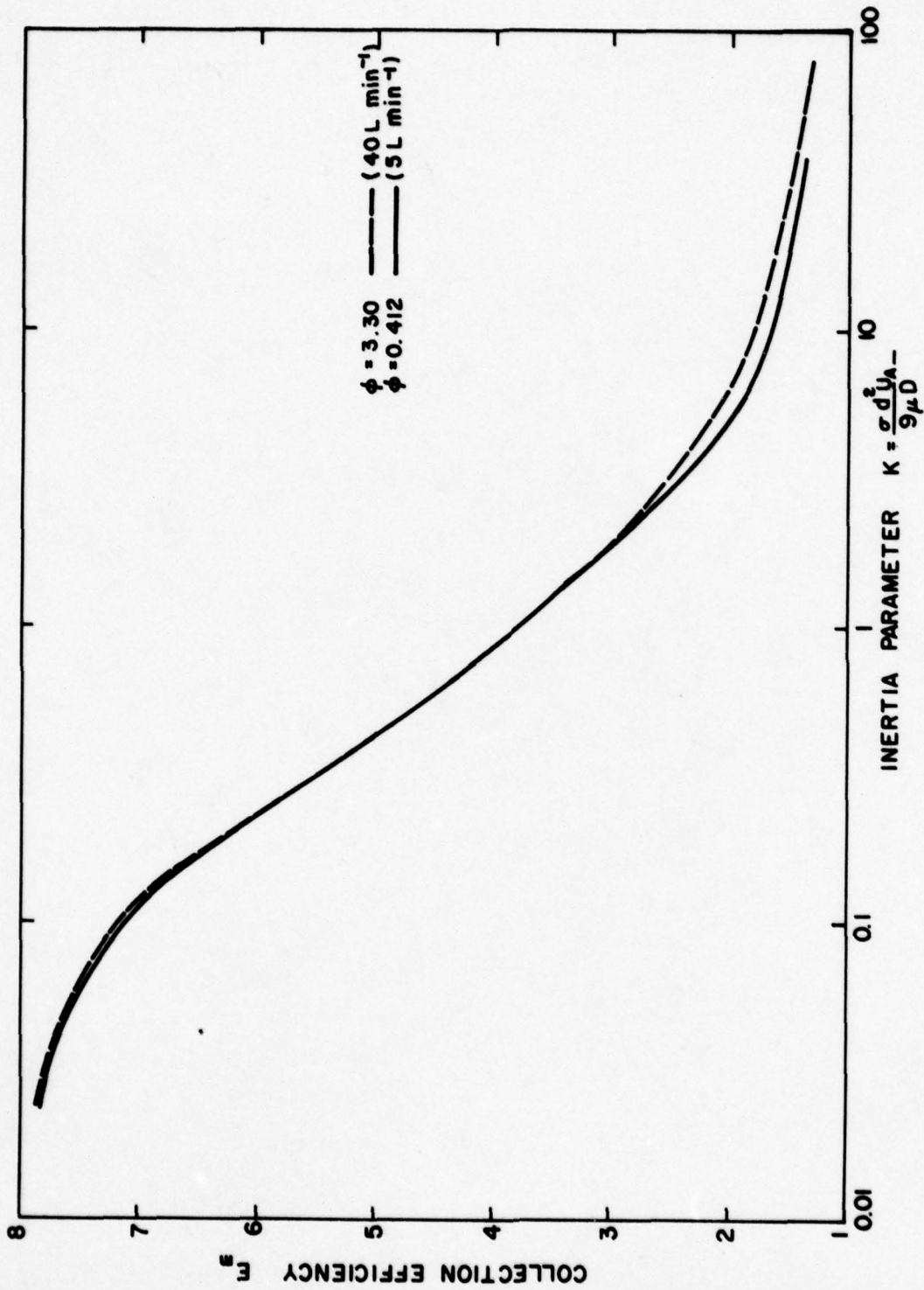


FIGURE 13: EFFECT OF FLOW VELOCITY IN SAMPLING TUBE ON COLLECTION EFFICIENCY FOR FIXED VELOCITY RATIO ($\frac{U_a}{U_a} = \frac{1}{10}$)

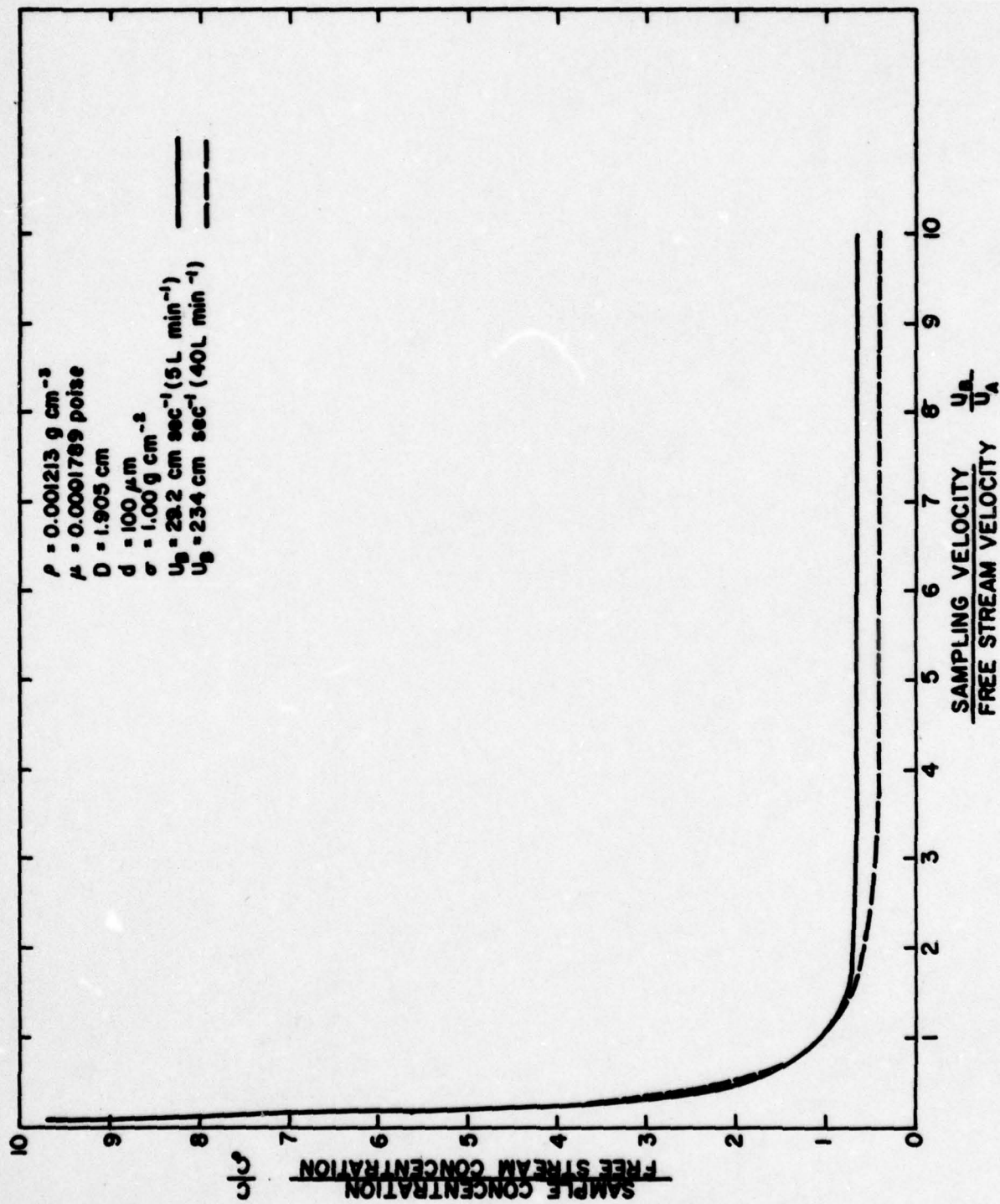


FIGURE 14: EFFECT OF FREE STREAM VELOCITY ON SAMPLING EFFICIENCY FOR ONE PARTICLE SIZE AND CONSTANT LOW VELOCITY IN SAMPLING TUBE

APPENDIX A

COMPUTER PROGRAM FOR CALCULATING THE STREAM FUNCTION

PAGE 1

// JOB T 0108

LOG DRIVE CART SPEC CART AVAIL PHY DRIVE
0000 0108 0000

V2 M11 ACTUAL 16K CONFIG 16K

// FOR
*ONE WORD INTEGERS
*LIST ALL

SUBROUTINE REPSI
COMMON PSI(61,89),ITERS,URAT,UA,RS,NPRB,NPZB,UB,RA,NPZ,NPR
COMMON INTVL,INDEX,ITER
READ(1,1)PSI,ITERS,URAT,UA,RS,NPRB,NPZB,UB,RA,NPZ,NPR
WRITE(3,200)
WRITE(3,201)RA,RS,NPZ,NPR,NPZB,NPRB,URAT,UA,UB,ITERS
WRITE(3,202)

DO 24 J=1,NPZ,INTVL
24 WRITE(3,203)(PSI(I,J),I=1,NPR,INTVL)
200 FORMAT(20H1 DISK STORAGE CHECK)
201 FORMAT(65H0STREAM FUNCTION FOR FLOW IN TWO CONCENTRIC PIPES WITH P
1PARAMETERS/ 10HORA = ,F7.2/ 10H RB = ,F7.2/ 10H NPZ = ,
215 / 10H NPR = ,15/ 10H NPZB = ,15/ 10H NPRB = ,15 /
310H URAT = ,F9.4/10H UA = ,F9.4/10H UB = ,F9.4 /
410H ITERS = ,15)
202 FORMAT(45H1THE CURRENT VALUES OF PSI STORED ON DISK ARE)
203 FORMAT('0',16F7.4)
RETURN
END

VARIABLE ALLOCATIONS

PSI(RC)=7FFE-5596 ITERS(IC)=5595
NPZ(RC)=558C UB(RC)=558A
INDEX(IC)=5584 ITER(IC)=5583

URAT(RC)=5592
RA(RC)=5588
J(I)=0000

UA(RC)=5590
NPZ(IC)=5587
I(I)=0001

RB(RC)=558E
NPR(IC)=5586

NPRB(IC)=558D
INTVL(IC)=5585

STATEMENT ALLOCATIONS

200 =0006 201 =0012 202 =0085 203 =009E 24 =00E2

FEATURES SUPPORTED
ONE WORD INTEGERS

CALLED SURPROGRAMS
SWRT SCOMP SIOFX SIOF SIOI SUBSC SDRED SDAF SDF SDI

INTEGER CONSTANTS
1=0004 3=0005

CORE REQUIREMENTS FOR REPSI
COMMON 10878 VARIABLES 4 PROGRAM 262

PAGE 3

// FOR
*ONE WORD INTEGERS
*LIST ALL
C

C THIS SUBROUTINE CALCULATES THE STREAM FUNCTION FOR FLOW THROUGH
C TWO CONCENTRIC PIPES WITH A TAPERED OUTSIDE WALL ON THE
C INSIDE PIPE
C

C SUBROUTINE SBM24
C COMMON PSI(61,89),ITERS,URAT,UA,UB,NPRB,NPZB,UB,RA,NPZ,NPR
C COMMON INTVL,INDEX,ITER

C READ AND CHECK INPUT PARAMETERS
C

C READ(2,100)NZ,NR,ITMAX,EPSMX
C READ(2,101)RA,RB,XL,D,UA,UB
C WRITE(3,200)NZ,NR,ITMAX,EPSMX,RA,RR,XL,D,UA,UB,INTVL,ITER

C CALCULATE AND WRITE DIMENSIONLESS PARAMETERS
C

C ALPHA=RB/RA
C BETA=XL/RA
C GAMMA=D/RA
C DELTA = (BETA-GAMMA)/ALPHA
C UC=(UA-ALPHA**2*UB)/(1.-((RB+2.0)/RB)*ALPHA)**2)
C UCRA=UC/UA
C PSIR=URAT*ALPHA**2
C WRITE(3,201)ALPHA,BETA,GAMMA,DELTA,URAT,UCRA

C R=RD/RA, Z=ZD/RA, PSI=PSID/((1.0/2.0)*UA**2)

C ESTABLISH BOUNDARY POINTS
C

C NPZ=NZ+1
C NPR=NR+1
C DELR=1.0/FLOAT(NR)
C XZR=((XL-D)/XL)*FLOAT(NZ)
C NZB=IFIX(XZB + 0.1)
C NPZR=NZB+1

C NPZS=NPZR+12
C XRB=ALPHA*FLOAT(NR)
C NRB=IFIX(XRB + 0.1)
C NRR=NRB+1
C NPRS = NPRB+1
C NPRC = NPRB + 2

PAGE 4

IF PSI IS PARTIALLY CALCULATED AND IN FILE
GO DIRECTLY TO FURTHER ITERATIONS

IF(INDEX)1,1,9
1 CONTINUE

ESTABLISH INITIAL GUESSES FOR STREAM FUNCTION
AND SET BOUNDARY CONDITIONS ON CENTRE LINE
AND INLET OF OUTSIDE PIPE

DO 2 I=1,NR
RI=FLOAT(I-1)
PSI(I,1)=(RI*DELRI)**2
DO 2 J=2,NPZ
2 PSI(I,J)=0.0

SET BOUNDARY CONDITION AT OUTLET OF INSIDE PIPE

DO 3 I = 2,NPRB
RI=FLOAT(I-1)
3 PSI(I,NPZ) = URAT*(RI*DELRI)**2

SFT BOUNDARY CONDITION AT OUTLET OF OUTSIDE PIPE

DO 4 I = NPRC,NR
RI=FLOAT(I-1)
4 PSI(I,NPZ)=1.0-UCRA*(1.0-(RI*DELRI)**2)

SET BOUNDARY CONDITION AT INSIDE WALL OF INSIDE PIPE

DO 5 J=NPZB,NPZ
5 PSI(NPRB,J)=PSIR

SET BOUNDARY CONDITION AT WALL OF OUTSIDE PIPE

DO 6 J=1,NPZ
6 PSI(NPR,J)=1.0

POINTS WITHIN PIPE MATERIAL ON FIRST GRID LINE
AFTER INSIDE PIPE RADIUS

DO 7 J = NPZB,NPZ
7 PSI(NPRS,J) = PSIR

COMPUTE SUCCESSIVELY BETTER APPROXIMATIONS FOR

PAGE 5

C THE STREAM FUNCTION AT ALL GRID POINTS, ITERATING BY
C THE GAUSS-SEIDEL METHOD UNTIL THE CONVERGENCE CRITERION
C IS SATISFIED
C

8 EPS=0.0
ITER=ITER+1
ITERS = ITER
DO 70 I=2,NR
DO 70 J=2,NZ

C INTERIOR POINTS FROM CENTRE LINE TO INSIDE PIPE RADIUS
C

15 CONTINUE
IF(I-NPRB)15,16,20
HOLDT=PSI(I,J)
PSI(I,J)=(PSI(I,J+1)+PSI(I,J-1)+PSI(I+1,J)+PSI(I-1,J))/4.0
1-(PSI(I+1,J)-PSI(I-1,J))/(18.0*FLOAT(I-1))
EPS=EPS+ABS(PSI(I,J)-HOLDT)
GO TO 70

C GRID LINE ON INSIDE PIPE RADIUS
C

16 CONTINUE
IF (J - NPZB)17,18,18
17 CONTINUE
HOLDT=PSI(I,J)
PSI(I,J)=(PSI(I,J+1)+PSI(I,J-1)+PSI(I+1,J)+PSI(I-1,J))/4.0
1-(PSI(I+1,J)-PSI(I-1,J))/(18.0*FLOAT(I-1))
EPS=EPS+ABS(PSI(I,J)-HOLDT)

18 CONTINUE
GO TO 70
20 CONTINUE

C FIRST GRID LINE AFTER INSIDE PIPE RADIUS
C

25 IF(I-NPRB-1)70,25,30
26 CONTINUE
HOLDT=PSI(I,J)
PSI(I,J)=(PSI(I,J+1)+PSI(I,J-1)+PSI(I+1,J)+PSI(I-1,J))/4.0
1-(PSI(I+1,J)-PSI(I-1,J))/(18.0*FLOAT(I-1))
EPS=EPS+ABS(PSI(I,J)-HOLDT)
GO TO 70

27 CONTINUE
IF(J-NPZB-1)28,30,30
28 CONTINUE

PAGE 6

C
C POINTS FOR WHICH INTERPOLATION IS REQUIRED
C ON FIRST GRID LINE AFTER INSIDE PIPE RADIUS
C

HOLDT=PSI(I,J)
A=FLOAT(NPZ-12-J)/12.0
XK1=A/(2.0*(A+1.0))
XK2=PSI(I,J-1)+PSI(I,J+1)
XK3=2.0*PSIB/(A*(A+1.0))
XK4=2.0*PSI(I+1,J)/(A+1.0)
XK5=(PSIB-PSI(I+1,J))/(FLOAT(I-1)*(A+1.0))
PSI(I,J)=(XK2+XK3+XK4+XK5)*XK1
EPS=EPS+ABS(PSI(I,J)-HOLDT)
GO TO 70

30 CONTINUE

C
C SECOND GRID LINE FROM INSIDE PIPE RADIUS

IF(I-NPRB-2)70,35,40
35 CONTINUE
IF(J-NPZB-12)36,36,37
36 CONTINUE

HOLDT=PSI(I,J)
PSI(I,J)=(PSI(I,J+1)+PSI(I,J-1)+PSI(I+1,J)+PSI(I-1,J))/4.0
1-(PSI(I+1,J)-PSI(I-1,J))/(8.0*FLOAT(I-1))
EPS=EPS+ABS(PSI(I,J)-HOLDT)
GO TO 70

37 CONTINUE

C
C POINTS FOR WHICH INTERPOLATION IS REQUIRED
C ON SECOND GRID LINE AFTER INSIDE PIPE RADIUS

HOLDT=PSI(I,J)
A=FLOAT(NPZ-J)/12.0
XK1=A/(2.0*(A+1.0))
XK2=PSI(I,J-1)+PSI(I,J+1)
XK3=2.0*PSIB/(A*(A+1.0))
XK4=2.0*PSI(I+1,J)/(A+1.0)
XK5=(PSIB-PSI(I+1,J))/(FLOAT(I-1)*(A+1.0))
PSI(I,J)=(XK2+XK3+XK4+XK5)*XK1
EPS=EPS+ABS(PSI(I,J)-HOLDT)
GO TO 70

40 CONTINUE

C
C INTERIOR POINTS BETWEEN INSIDE AND OUTSIDE PIPE

C

PAGE 8

```

110M UR = .F7.2/ 10M INTVL = .15/ 10M ITER = .15)
201 FORMAT(37#CALCULATED DIMENSIONLESS PARAMETERS /
110#ALPHA = .F9.4/ 10M BETA = .F9.4/ 10 M GAMMA = .F9.4/
1 10M DELTA = .F9.4/ 10M URAT = .F9.4/ 10M UCRA = .F9.4)
202 FORMAT(4#)CONVERGENCE CONDITION HAS BEEN REACHED AFTER
115. 11H ITERATIONS/ 38H THE STREAM FUNCTION FIELD IS GIVEN BY)
203 FORMAT(1H0, 16F7.4)
204 FORMAT(42#)NO CONVERGENCE. CURRENT VALUES OF PSI ARE)
205 FORMAT(26#) THE ITERATION COUNTER IS.15,
134H AND THE CURRENT VALUES OF PSI ARE )
206 FORMAT(20# DISK WRITE COMPLETE)
300 RETURN

```

END

VARIABLE ALLOCATIONS

```

PSI(RC)=7FFE-5596 ITERS(IC)=5595 URAT(RC)=5592 UA(RC)=5590 RB(RC)=558E NPRB(IC)=558D
NPZB(IC)=558C UB(RC)=558A RA(RC)=5588 EPSMX(R)=0000 DELTAIR)=000C XRB(R)=0018 XK2(R)=0024 ITMAX(I)=0034 J(I)=003B
INDEX(IC)=5584 GAMMA(R)=000A XZB(R)=0016 XK1(R)=0022 NR(I)=0033 NPRC(I)=0039 I(I)=003A
RETA(R)=0008 DELR(R)=0014 A(R)=0020 NZ(I)=0032 NPS(I)=0036 ITD(I)=003C
ALPHA(R)=0010 PSIB(R)=0012 HOLDT(R)=001E XK3(R)=0026 NPB(I)=0035 J(I)=003B
UCRA(R)=0010 EPSIR)=001C XK4(R)=0028 NPZ(I)=0036 ITD(I)=003C
ALPHAIR)=0006 PSIBIR)=0012 HOLDTIR)=001E XK5IR)=002A NRB(I)=0037

```

STATEMENT ALLOCATIONS

```

100 =0057 101 =005F 200 =0062 201 =00E5 202 =C12A 203 =C160 204 =C165 205 =017C 206 =019E 1 =029A
2 =02RA 3 =02E6 4 =030F 5 =0331 6 =0349 7 =0361 8 =0375 15 =0393 16 =03FD 17 =0403
14 =0468 20 =046D 25 =0477 26 =047D 27 =04E7 28 =04EF 30 =0591 35 =0598 36 =05A3 37 =060D
40 =06AD 70 =0715 74 =0737 75 =0741 80 =0785 82 =078C 83 =079E 85 =07C5 86 =07CD
300 =07F2

```

FEATURES SUPPORTED
ONE WORD INTEGERS

CALLED SUBPROGRAMS

```

FABS FADD FADDX FSUB FSUBX FMPY FMPYX FDIV FLD FLDX FSTO FSTOX FSBR FSBR SDF FAXI
IFIX FLOAT SRED SREDX SWRT SWRTX SIOF SIOFX SIOF SIOI SIOF SIOI SDF SDF SDF SDF SDF SDF SDF SDF SDF SDF

```

REAL CONSTANTS

```

.100000E 01=0044 .200000E 01=0046 .100000E 00=0048 .000000E 00=004A .400000E 01=004C .800000E 01=004E
.120000E 02=0050

```

INTEGER CONSTANTS

```

2=0052 3=0053 1=0054 12=0055 100=0056

```

```

CORE REQUIREMENTS FOR SBM24
COMMON 10878 VARIABLES 68 PROGRAM 1968

```


PAGE 9

RELATIVE ENTRY POINT ADDRESS IS 01AA (HEX)

END OF COMPILATION

// DUP

*STORE WS UA SBM24
CART ID 0108 DB ADDR 4C12 DB CNT 0088

// EJECT

SECTOPEP 10000000

100 *DUPC 10000000 10000000 10000000 10000000

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```
// FOR
*ONE WORD INTEGERS
*IOCS(CARD)
*IOCS(1132 PRINTER)
*IOCS(DISK)
*LIST ALL
DEFINE FILE 1(5439,2,U,18)
COMMON PSI(61,89),ITERS,URAT,UA,UB,RA,NPRB,NPZB,UB,RA,NPZ,NPR
COMMON INTVL,INDEX,ITER
READ(2,100) ITER,INTVL
IF(ITER)1,1,2
1 INDEX=-1
CALL SBW24
WRITE(1,1)PSI,ITERS,URAT,UA,UB,RA,NPRB,NPZB,UB,RA,NPZ,NPR
GO TO 3
2 INDEX =1
CALL REPSI
CALL SBW24
WRITE(1,1)PSI,ITERS,URAT,UA,UB,RA,NPRB,NPZB,UB,RA,NPZ,NPR
3 CONTINUE
100 FORMAT(2I5)
CALL EXIT
END
```

VARIABLE ALLOCATIONS

```
PSI(IRC)=7FFE=5596 ITERS(1C)=5595 URAT(IRC)=5592 UA(IRC)=5590 RB(IRC)=558E NPRB(1C)=558D
NPZ(1C)=558C UB(IRC)=558A RA(IRC)=5588 NPZ(1C)=5587 NPR(1C)=5586 INTVL(1C)=5585
INDEX(1C)=5584 ITER(1C)=5583 IB(1 )=0008
```

STATEMENT ALLOCATIONS

```
100 =000C 1 =0033 2 =0057 3 =007A
```

FEATURES SUPPORTED
ONE WORD INTEGERS
IOCS

CALLED SURPROGRAMS

```
SRM24 RFPSSI FLD FSTO CARDZ PRNTZ SRED SF10 SF10I SDF10 SDWRT SDCOM SDAF SDF SD1
INTEGER CONSTANTS
2=000A 1=000B
```

CORE REQUIREMENTS FOR

```
COMMON 10878 VARIABLES 10 PROGRAM 114
```

END OF COMPILATION

PAGE 11

// XFO 1

*FILES(1,PSI10)

11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM
11/10/82 11:10 AM

11/10/82 11:10 AM

11/10/82 11:10 AM

DISK STORAGE CHECK

STREAM FUNCTION FOR FLOW IN TWO CONCENTRIC PIPES WITH PARAMETERS

RA = 60.00
RR = 12.00
NP7 = 89
NPR = 61
NPZR = 65
NPRR = 13
URAT = 3.0000
UA = 1.0000
UR = 3.0000
ITFRS = 2800

THE CURRENT VALUES OF PSI STORED ON DISK ARE

0.0000	0.0044	0.0177	0.0399	0.0711	0.1111	0.1599	0.2177	0.2844	0.3599	0.4444	0.5377	0.6399	0.7511	0.8711	1.0000
0.0000	0.0044	0.0178	0.0400	0.0712	0.1113	0.1602	0.2181	0.2848	0.3603	0.4448	0.5381	0.6402	0.7513	0.8712	1.0000
0.0000	0.0044	0.0178	0.0402	0.0714	0.1115	0.1606	0.2184	0.2852	0.3608	0.4452	0.5384	0.6405	0.7515	0.8713	1.0000
0.0000	0.0044	0.0179	0.0403	0.0716	0.1118	0.1609	0.2189	0.2857	0.3613	0.4457	0.5389	0.6409	0.7517	0.8714	1.0000
0.0000	0.0045	0.0180	0.0404	0.0719	0.1122	0.1614	0.2194	0.2862	0.3618	0.4462	0.5394	0.6413	0.7520	0.8716	1.0000
0.0000	0.0045	0.0181	0.0407	0.0722	0.1127	0.1620	0.2201	0.2869	0.3625	0.4469	0.5400	0.6418	0.7524	0.8718	1.0000
0.0000	0.0045	0.0182	0.0409	0.0727	0.1133	0.1627	0.2209	0.2878	0.3634	0.4477	0.5407	0.6424	0.7528	0.8720	1.0000
0.0000	0.0046	0.0184	0.0413	0.0733	0.1141	0.1637	0.2220	0.2889	0.3645	0.4487	0.5416	0.6431	0.7533	0.8722	1.0000
0.0000	0.0046	0.0186	0.0418	0.0741	0.1152	0.1650	0.2234	0.2904	0.3659	0.4500	0.5427	0.6440	0.7539	0.8726	1.0000
0.0000	0.0047	0.0190	0.0426	0.0752	0.1167	0.1667	0.2252	0.2922	0.3676	0.4516	0.5440	0.6450	0.7546	0.8729	1.0000
0.0000	0.0049	0.0196	0.0437	0.0769	0.1187	0.1690	0.2276	0.2945	0.3697	0.4534	0.5456	0.6462	0.7555	0.8734	1.0000
0.0000	0.0051	0.0205	0.0454	0.0793	0.1216	0.1721	0.2306	0.2973	0.3723	0.4556	0.5474	0.6476	0.7565	0.8739	1.0000
0.0000	0.0055	0.0219	0.0481	0.0829	0.1256	0.1761	0.2344	0.3008	0.3753	0.4582	0.5495	0.6492	0.7575	0.8744	1.0000
0.0000	0.0062	0.0243	0.0523	0.0883	0.1313	0.1814	0.2392	0.3049	0.3788	0.4610	0.5518	0.6510	0.7587	0.8750	1.0000
0.0000	0.0073	0.0284	0.0597	0.0968	0.1391	0.1881	0.2447	0.3095	0.3826	0.4641	0.5542	0.6528	0.7599	0.8757	1.0000
0.0000	0.0090	0.0355	0.0737	0.1101	0.1492	0.1958	0.2508	0.3144	0.3865	0.4673	0.5567	0.6547	0.7612	0.8763	1.0000
0.0000	0.0110	0.0456	0.1199	0.1274	0.1599	0.2035	0.2567	0.3191	0.3904	0.4704	0.5591	0.6565	0.7624	0.8769	1.0000
0.0000	0.0124	0.0512	0.1199	0.1372	0.1679	0.2098	0.2619	0.3233	0.3939	0.4733	0.5614	0.6582	0.7636	0.8775	1.0000
0.0000	0.0130	0.0527	0.1199	0.1398	0.1720	0.2141	0.2658	0.3268	0.3969	0.4759	0.5635	0.6599	0.7647	0.8781	1.0000
0.0000	0.0132	0.0531	0.1199	0.1398	0.1737	0.2165	0.2684	0.3294	0.3994	0.4780	0.5654	0.6613	0.7657	0.8786	1.0000
0.0000	0.0133	0.0532	0.1199	0.1388	0.1739	0.2177	0.2701	0.3314	0.4013	0.4799	0.5670	0.6626	0.7667	0.8791	1.0000
0.0000	0.0133	0.0533	0.1199	0.1372	0.1735	0.2181	0.2712	0.3328	0.4029	0.4815	0.5684	0.6638	0.7675	0.8796	1.0000
0.0000	0.0133	0.0533	0.1199	0.1355	0.1727	0.2182	0.2720	0.3340	0.4043	0.4829	0.5698	0.6649	0.7683	0.8800	1.0000

STREAM FUNCTION FOR FLOW IN TWO CONCENTRIC PIPES WITH PARAMETERS

NZ = PR
NR = 60
ITMAX = 5000
EPSMX = 0.10E-02
RA = 60.00
RR = 12.00
XL = 84.00
D = 24.00
UA = 1.00
UR = 3.00
INTVL = 4
ITFR = 2800

CALCULATED DIMENSIONLESS PARAMETERS

ALPHA = 0.2000
BETA = 1.4666
GAMMA = 0.4000
DELTA = 5.3333
URAT = 3.0000
UCRA = 0.9306

CONVERGENCE CONDITION HAS BEEN REACHED AFTER 2890 ITERATIONS
 THE STREAM FUNCTION FIELD IS GIVEN BY

0.0000	0.0044	0.0177	0.0399	0.0711	0.1111	0.1599	0.2177	0.2844	0.3599	0.4444	0.5377	0.6399	0.7511	0.8711	1.0000
0.0000	0.0044	0.0178	0.0401	0.0712	0.1113	0.1602	0.2181	0.2848	0.3604	0.4448	0.5381	0.6402	0.7513	0.8712	1.0000
0.0000	0.0044	0.0178	0.0402	0.0714	0.1116	0.1606	0.2185	0.2852	0.3608	0.4452	0.5385	0.6406	0.7515	0.8713	1.0000
0.0000	0.0044	0.0179	0.0403	0.0716	0.1119	0.1610	0.2189	0.2857	0.3613	0.4457	0.5389	0.6409	0.7517	0.8714	1.0000
0.0000	0.0045	0.0180	0.0405	0.0719	0.1122	0.1614	0.2194	0.2863	0.3619	0.4462	0.5394	0.6413	0.7520	0.8716	1.0000
0.0000	0.0045	0.0181	0.0407	0.0722	0.1127	0.1620	0.2201	0.2870	0.3626	0.4469	0.5400	0.6418	0.7524	0.8718	1.0000
0.0000	0.0045	0.0182	0.0409	0.0727	0.1133	0.1627	0.2209	0.2879	0.3635	0.4478	0.5407	0.6424	0.7528	0.8720	1.0000
0.0000	0.0046	0.0184	0.0413	0.0733	0.1141	0.1637	0.2220	0.2890	0.3646	0.4488	0.5416	0.6431	0.7533	0.8723	1.0000
0.0000	0.0046	0.0186	0.0419	0.0741	0.1152	0.1650	0.2234	0.2904	0.3660	0.4501	0.5427	0.6440	0.7540	0.8726	1.0000
0.0000	0.0047	0.0190	0.0426	0.0753	0.1167	0.1667	0.2253	0.2923	0.3677	0.4516	0.5441	0.6451	0.7547	0.8730	1.0000
0.0000	0.0049	0.0196	0.0437	0.0769	0.1188	0.1690	0.2276	0.2945	0.3698	0.4535	0.5456	0.6463	0.7555	0.8734	1.0000
0.0000	0.0051	0.0205	0.0454	0.0793	0.1216	0.1721	0.2307	0.2974	0.3724	0.4557	0.5474	0.6477	0.7565	0.8739	1.0000
0.0000	0.0055	0.0219	0.0481	0.0829	0.1257	0.1762	0.2345	0.3008	0.3754	0.4582	0.5499	0.6493	0.7576	0.8744	1.0000
0.0000	0.0062	0.0243	0.0523	0.0883	0.1313	0.1815	0.2392	0.3049	0.3788	0.4611	0.5518	0.6510	0.7587	0.8750	1.0000
0.0000	0.0073	0.0284	0.0597	0.0968	0.1392	0.1881	0.2448	0.3095	0.3826	0.4642	0.5542	0.6528	0.7600	0.8757	1.0000
0.0000	0.0080	0.0355	0.0737	0.1101	0.1493	0.1958	0.2508	0.3144	0.3866	0.4674	0.5567	0.6547	0.7612	0.8763	1.0000
0.0000	0.0110	0.0456	0.1199	0.1274	0.1600	0.2036	0.2568	0.3191	0.3904	0.4705	0.5592	0.6565	0.7625	0.8769	1.0000
0.0000	0.0124	0.0512	0.1199	0.1372	0.1679	0.2099	0.2619	0.3234	0.3939	0.4733	0.5615	0.6583	0.7636	0.8776	1.0000
0.0000	0.0130	0.0527	0.1199	0.1398	0.1720	0.2141	0.2658	0.3268	0.3969	0.4759	0.5636	0.6599	0.7647	0.8781	1.0000
0.0000	0.0132	0.0531	0.1199	0.1398	0.1737	0.2165	0.2685	0.3295	0.3994	0.4781	0.5654	0.6613	0.7657	0.8786	1.0000
0.0000	0.0133	0.0532	0.1199	0.1388	0.1739	0.2177	0.2702	0.3314	0.4013	0.4799	0.5670	0.6626	0.7667	0.8791	1.0000
0.0000	0.0133	0.0533	0.1199	0.1373	0.1735	0.2181	0.2712	0.3328	0.4029	0.4815	0.5684	0.6638	0.7675	0.8796	1.0000
0.0000	0.0133	0.0533	0.1199	0.1355	0.1727	0.2182	0.2720	0.3340	0.4043	0.4829	0.5698	0.6649	0.7683	0.8800	1.0000

APPENDIX B

COMPUTER PROGRAM FOR SOLVING THE EQUATIONS OF MOTION

```

PAGE 1
// JOB T
LOG DRIVE   CAPT SPEC   CART AVAIL   PHY DRIVE
0000        010R        0108        0000
          4000        4000        0001

V2 M11   ACTUAL 16K   CONFIG 16K

// FOR
*ONE WORD INTEGERS
*LIST ALL
SUBROUTINE GTPSI
C
C THIS SUBROUTINE RETRIEVES THE STREAM FUNCTION ARRAY AND ASSOCIATED
C   CONSTANTS FROM DISK
C
COMMON PSI(61,80),URAT,RR,NPRR,NPZR,RA,APR,INTVL
READ(11)PSI
READ(11)5430)ITERS,URAT,UA,RR,NPRR,NPZR,UP,RA,NPZ,NPR
WRITE(3,200)
WRITE(3,201)RA,RR,NPZ,NPR,NPZR,NPRR,URAT,UA,UR,ITERS
WRITE(3,202)
NPZMO = NPZ - 9
DO 24 J=1,NPZMO,INTVL
24 WRITE(3,203)PSI(I,J),I=1,NPR,INTVL)
200 FORMAT(20HC DISK STORAGE CHECK)
201 FORMAT(65H0STREAM FUNCTION FOR FLOW IN TWO CONCENTRIC PIPES WITH P
18PARAMETERS/ 10H0RA = ,F7.2/ 10H RR = ,F7.2/ 10H NPZ = ,
215 / 10H NPR = ,I5/ 10H NPZR = ,I5/ 10H NPRR = ,I5 /
410H URAT = ,F9.4/10H UA = ,F9.4/10H UR = ,F9.4 /
410H ITERS = ,I5)
202 FORMAT(45H1THE CURRENT VALUES OF PSI STORED ON DISK ARE)
203 FORMAT(10,16F7.4)
RETURN
END

VARIABLE ALLOCATIONS
PSI(19C)=7EFF-59F0   URAT(1C)=59DE   RB(1C)=59DC   NPZB(1C)=59DA   RA(1C)=59D8
NPR(1C)=59D7         INTVL(1C)=59D6   UA(1R)=0000   UR(1R)=0002   NPZ(1R)=0004
NPZMO(1I)=0006      J(1I)=0007      I(1I)=0008   ITERS(1I)=0004

STATEMENT ALLOCATIONS
200 =000E 201 =001A 202 =008D 203 =00A6 24 =00F3

FEATURES SUPPORTED
ONE WORD INTEGERS

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PAGE 2

CALLD SURPROGRAMS
SWRT SCOMP SIOFX SIOF SIOI SUBSC SDRED SDAF SDF SDI

INTEGER CONSTANTS
1=000A 5430=000B 3=000C 9=000D

CORE REQUIREMENTS FOR GTPSI
COMMON 9770 VARIABLES 10 PROGRAM 272

RELATIVE ENTRY POINT ADDRESS IS 00AB (HEX)

END OF COMPILATION

// DUP

*STORE WS UA GTPSI
CART ID 0108 DR ADDR 4E20 DB CNT 0C13

// FJECT

PAGE 3

// FOR
*ONE WORD INTEGERS
*LIST ALL

FUNCTION CORE(RE)

THIS FUNCTION COMPUTES THE PRODUCT OF DRAG COEFFICIENT
AND REYNOLDS NUMBER FOR A SPHERE AS A FUNCTION OF
REYNOLDS NUMBER

CONSTANT COEFFICIENTS

A1=1.724.
A2=-2.9263*1.E-04
A3=2.0154*1.E-06
A4=-6.9105*1.E-09
R0=-1.29536
R1=0.86*1.E-01
R2=-4.6677*1.E-02
R3=1.1225*1.E-03

CHOOSE THE APPROPRIATE POLYNOMIAL

IF (RE-6.0)2,7,7

INITIAL ESTIMATE

2 IF (RE-0.00001)3,4,4
3 CORE = 24.0
GO TO 3
4 X=24.*RF

BEGIN NEWTON METHOD ITERATION

CONTINUE

DO 6 ITER=1,20

FX=A1*X+A2*X**2+A3*X**3+A4*X**4-RF

FPX=A1+2.*A2*X+3.*A3*X**2+4.*A4*X**3

DELX=FX/FPX

X=X-DELX

CHECK FOR CONVERGENCE

FPS=1.E-06

IF (ABS(DELX/X))-FPS)5,5,6

5 CORE=X/RF

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PAGE 4

GO TO 30
6 CONTINUE
GO TO 29

C INITIAL ESTIMATE

7 CD = 1.0
FLOG = 0.634294481903252
X=ALOG(CD*RE**2)*ELOG

C BEGIN NEWTON METHOD ITERATION

DO 24 ITR=1,20
FX=R0+R1*X+R2*X**2+R3*X**3 - ALOG(RE)*ELOG
FDX=R1+2.*R2*X+3.*R3*X**2
DFLX=FX/FPX
X=X-DELX

C CHECK FOR CONVERGENCE

EPS=1.F-06
IFIARS(DELX/X)-EPS)22,22,24
22 CONTINUE
GO TO 30

24 CONTINUE
20 WRITE(3,207)
30 RETURN

C FORMATS FOR OUTPUT STATEMENTS

207 FORMAT(16H0 NO CONVERGENCE)

C END

C VARIABLE ALLOCATIONS

CDRE(R)=0000
R1(R)=0000
DFLX(R)=001R

A1(R)=0002
B2(R)=000E
EPS(R)=001A

A2(R)=0004
B3(R)=0010
CD(R)=001C

A3(R)=0006
X(R)=0012
ELOG(R)=001E

A4(R)=0008
FX(R)=0014
ITER(I)=0028

B0(R)=000A
FPX(R)=0016

C STATEMENT ALLOCATIONS

202 =0059 2 =00A8 3 =00AF 4 =00B5 5 =012D 6 =0135 7 =0140 22 =0188 24 =01C3 29 =01CC
30 =01DD

C FEATURES SUPPORTED
0NF WORD INTFGERS

C CALLED SUPROGRAMS
FAPF FALOG FAXR FADD FSUR FMPY FDIV FLD FSTO FSBR FAXI SWRT SCOMP SNR SUBIN

PAGE 5

REAL CONSTANTS

.100000F 01=002A .240000F 02=002C .233630E 01=002E .100000E-03=0030 .201540E 01=0032 .100000E-05=0034
.691050F 01=0036 .100000E-08=003H .129536E 01=003A .986000E 01=003C .100000E 00=003E .466770E 01=0040
.100000F-01=0042 .112350F 01=0044 .100000E-02=0046 .400000E 01=0048 .100000E-04=004A .200000E 01=004C
.100000F 01=004F .434294F 00=0050 .100000E 02=0052

INTEGER CONSTANTS

1=0054 20=0055 2=0056 3=0057 4=0058

CODE REQUIREMENTS FOR CORE

COMMON 0 VARIABLES 42 PROGRAM 426

RELATIVE ENTRY POINT ADDRESS IS 0053 (HEX)

END OF COMPILATION

// DUP

*STORE WS UA CDRE
CARD ID 0108 DR ADDR 4E33 DR CNT 0022

// EJECT

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PAGE 6

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// FOR
*ONF WORD INTEGERS
*LIST ALL
SUBROUTINE SRM22(N,Y,F,X,H,IRUNG,M)
C
C FOURTH ORDER RUNGE KUTTA METHOD
C FOR N FIRST ORDER O.D.E.
C DIMENSION PHI(50),SAVY(50),Y(50),F(50)
C GO TO (2,3,4,5,6),Y
C
C PASS 1
C
C 2 IRUNG=1
C RETURN
C
C PASS 2
C
C 3 DO 22 J=1,N
C SAVY(J)=Y(J)
C PHI(J)=F(J)
C 22 Y(J)=SAVY(J)+0.5*HF(J)
C X=X+0.5*H
C IRUNG=1
C RETURN
C
C PASS 3
C 4 DO 23 J=1,N
C PHI(J)=PHI(J)+2.0*F(J)
C 33 Y(J)=SAVY(J)+0.5*HF(J)
C IRUNG=1
C RETURN
C
C PASS 4
C 5 DO 44 J=1,N
C PHI(J)=PHI(J)+2.0*F(J)
C 44 Y(J)=SAVY(J)+HF(J)
C X=X+0.5*H
C IRUNG=1
C RETURN
C PASS 5
C 6 DO 55 J=1,N
C 55 Y(J) = SAVY(J) + (PHI(J) + F(J))*H/6.0
C IRUNG=2
C RETURN
C
C END
```

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VARIABLE ALLOCATIONS
PMT(R 1=0062-0000 SAVY(R 1=00C6-0064 JCI 1=00C8

STATEMENT ALLOCATIONS
2 #0105 3 =0108 22 =011E 4 =0141 33 =0152 5 =016D 44 =017E 6 =019F 55 =01A3

FEATURES SUPPORTED
ONE WORD INTEGERS

CALLED SUBPROGRAMS

FADD FADXX EMPY EMPYX FDIY FLD FLDX FSTO FSTOX SUBSC SUBIN

REAL CONSTANTS

.500000F 0C=00CA .200000E 01=00CC .600000E 01=00CE

INTEGER CONSTANTS

1=0000 2=0001

CORP REQUIREMENTS FOR SRV22

COMMON 2 VARIABLES 2C2 PROGRAM 250

RELATIVE ENTRY POINT ADDRESS IS 00D2 (HEX)

END OF COMPILATION

// DUP

#STORE WS UA SRV22

CART ID 0108 DR ADDR 4E55 DB CNT 0012

// FJCT

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// FOR
*ONE WORD INTEGERS
*IOCS(ICARD)
*IOCS(I132 PRINTER)
*IOCS(DISK)
*LIST ALL

DIMENSION G(4),DG(4)
COMMON PSI(6),80),URAT,RR,NPRB,NPZR,RA,NPR,INTVL
DEFINE FILE I15439,2,U,19)
1 READ(2,100)G&LFT,G&RIT,SIGML,DTAU,NIRP,NSBP,NX
WRITE(3,200)
WRITE(3,201)G&LFT,G&RIT,SIGML,DTAU,NIRP,NSRP,NX

ESTABLISH PHYSICAL PROPERTIES FOR CALCULATING COLLECTION
EFFICIENCY

DC IS CYLINDER DIAMETER, CM
DP IS PARTICLE DIAMETER, CM
RHO IS FLUID DENSITY, GM/CC
SIGMA IS PARTICLE DENSITY, GM/CC
XMU IS ABSOLUTE VISCOSITY OF FLUID, POISE
UR IS FLUID VELOCITY IN INSIDE PIPE CM/SEC
ISR IS STARTING RATIO Z/RB = -G(3)

READ(2,101)DC,DP,RHO,SIGMA,XMU,UR,ISR

IF CORRECT STREAM FUNCTION VALUES ARE ALREADY IN TEMPORARY
STORAGE PASS BY RETRIEVAL FROM DISK STORAGE

READ(2,111) ICONF
IF(ICONF)2,3,3
2 CONTINUE

CALL STREAM FUNCTION ARRAY FROM DISK STORAGE
AND SET CONSTANTS FOR FLUID VELOCITY CALCULATION

READ(2,111) INTVL
CALL GTPSI
WRITE(3,211) INTVL
3 CONTINUE

UA IS FREE STREAM VELOCITY, CM/SEC

UA = UR/URAT
REZ=RHO*DP*UA/XMU
XK=SIGMA*DP*2*UA/(9.*XMU*DC)

PAGE 9

P = REZ**2/XK
WRITE(3,208)
WRITE(3,202) REZ,XK,P,DC,DP,RHO,SIGMA,XMU,UA,UB,ISR

C ESTABLISH GRID STEP SIZE

NR = NPR - 1
DELX = 1.0/FLOAT(NR)
FDRSQ = 4.0*DELX**2
RR=RR+0.001

C HALF INTERVAL ITERATION FOR INITIAL G4 VALUE

DO 21 ITER=1,NX

C SFT AND PRINT INITIAL CONDITIONS

M=0
NSTEP=0
TAU=0.0
GZER = -FLOAT(159)
G(3) = GZER
G4ZER=(G4LT+G4RT)/2.0
G(4)=G4ZER
I = 1+IFIX(RR*G(4))
JC=NPR-(NPR-1)*ISR
J = JC

R1=FLOAT(I-1)
UR=(PSI(I+1,J)-PSI(I-1,J))/(FDRSQ*RI)
UR=(PSI(I,J-1)-PSI(I,J+1))/(FDRSQ*RI)
G(1)=UZ
G(2)=UR
RF=REZ*(UR-G(2))**2+(UZ-G(1))**2**0.5

IP=ITER/NIRP*NIRP

IF(IP-ITER)5,7,5

5 CONTINUE

IF(ITER-1)6,7,6

6 CONTINUE

IF(ITER-NX)8,7,8

7 CONTINUE

WRITE(3,205)

WRITE(3,203)ITER,G4LT,G4ZER,G4RT,TAU,G(1),G(2),G(3),G(4),UZ,UR,

1XCODE

C CALL ON RUNGE KUTTA SUBROUTINE

C

PAGE 11

10 G4RIT=G4ZER
GO TO 21

20 G4LFT=G4ZER
21 CONTINUE

PRINT SOLUTIONS FOR FINAL VALUE OF TAU

TAU = TAU + 0.0001
WRITE(3,204)TAW,G(1),G(2),G(3),G(4),UZ,UR,XCDRE

CALCULATE THE COLLECTION EFFICIENCY

WRITE(3,209) G4ZER
FV = G4ZER**2
WRITE(3,206)EV

CALCULATE THE SAMPLING EFFICIENCY

RSINF=SQRT(PSI(NPRB,NPZB))*RA/RR
PSINT = PSI(NPRB,NPZB)
DO 25 I=1,NPR
IF(PSI(I,J0)-PSINT)25,25,24
24 RSINF = FLOAT(I-2)*SQRT(PSINT/PSI(I-1,J0))*DEL*RA/4B
GO TO 28

25 CONTINUE
28 CONTINUE

WRITE(3,210) RSINF
CR=(G4ZER/RSINF)**2
WRITE(3,207) CR
READ(2,111)NSTOP
IF(NSTOP)1,30,30
30 CALL EXIT

FORMATS FOR INPUT AND OUTPUT STATEMENTS

100 FORMAT(10X,F10.7,20X,F10.7,19X,F3.0/ 10X,F10.7,20X,I3,25X,I4/
1 9X,I3)

101 FORMAT(F10.5,F10.7,F10.6, F10.6,F10.7,F10.4,I5)

111 FORMAT(I5)
200 FORMAT(1H1, 37X, 40HCOLLECTION EFFICIENCY OF A CIRCULAR TURE/
1 1HC)

201 FORMAT(10H0.4LEF = ,F10.6/ 10H G4RIT = ,F10.6/10H SIGNL = ,
1 F4.0/ 10H DTAU = ,F10.6/ 10H NRP = ,I4/ 10H NSRP = ,I4/
2 10H NX = ,I4)

PAGE 12

202 FORMAT(10HOREZ = ,F12.7/10H XK = ,E12.6/
1 10H P = ,E10.4/
2 10HODC = ,F10.5/10H DP = ,F10.7/10H RMO = ,F10.6/
3 10H SIGMA = ,F10.6/10H XMU = ,F10.7/10H UA = ,F10.6/
4 10H UR = ,F10.4/10H ISR = ,F15)
203 FORMAT(10H0ITER = ,I3/ 10H G4LEF = ,F10.6/ 10H G4ZER = ,
1 F10.6/ 10H G4RIT = ,F10.6/ 7H0 TAU, 11X, 4HG(1), 12X,
2 4HG(2), 12X, 4HG(3), 12X, 4HG(4), 13X, 2H0Z, 14X, 2HUR ,
3 12X, 4HCDRE /
4 1M0, F7.4, 4F16.6, 3F16.4)
204 FORMAT (1H , F7.4, 4F16.6, 3F16.4)
205 FORMAT (44H1THE VCTION OF A CRITICAL PARTICLE IS GIVEN BY)
206 FORMAT(30HCTHE COLLECTION EFFICIENCY IS ,E10.4)
207 FORMAT(29HCTHE SAMPLING EFFICIENCY IS ,E10.4)
208 FORMAT(28H1THE PHYSICAL PARAMETERS ARE)
209 FORMAT(34HCTHE UPSTREA* PARTICLE RADIUS IS ,E10.4)
210 FORMAT(31HCTHE UPSTREA* FLUID RADIUS IS ,E10.4)
211 FORMAT(39HCTHE INTERVAL OF THE WRITTEN VALUES IS ,I5)

C

VARIABLE ALLOCATIONS

PSI(RC)=7FFF-59FC UPAT(RC)=59DE RB(RC)=59DC NPRB(IC)=59DB NPZB(IC)=59DA RAIRC)=59DB
MPR(IC)=59D7 INTVL(IC)=59D6 G(R)=000E-0008 DG(R)=0016-0010 G4LFT(R)=0018 G4RIT(R)=001A
SIGNL(R)=001C DTAU(R)=001E UA(R)=0020 DC(R)=0020 DPR(R)=0022 RHO(R)=0024 SIGMA(R)=0026
XMU(R)=0028 UB(R)=002A UR(R)=002C REZ(R)=002E XK(R)=0030 PIR(R)=0032
DEL(R)=0034 FDRSQ(R)=0036 RR(R)=0038 TAU(R)=003A G3ZER(R)=003C G4ZER(R)=003E
RIR(R)=0040 UZ(R)=0042 UR(R)=0044 RE(R)=0046 XCDRE(R)=0048 TAWIR(R)=004A
HITS(R)=004C EM(R)=004E RSNFR(R)=0050 PSIMT(R)=0052 CR(R)=0054 IB(I)=005C
NIPF(I)=005D NSRP(I)=005E NX(I)=005F ISR(I)=0060 ICONF(I)=0061 NR(I)=0062
ITFR(I)=0063 M(I)=0064 NSTEP(I)=0065 I(I)=0066 J(I)=0066
IP(I)=0069 IRUNG(I)=006A NSTOP(I)=006C

STATEMENT ALLOCATIONS

100 =00AD 101 =009F 111 =00A6 200 =00A8 201 =00C4 202 =00FC 203 =0154 204 =019C 205 =01A4 206 =01BD
207 =01CF 208 =01E1 209 =01F1 210 =0205 211 =0218 1 =0248 2 =0289 3 =0296 5 =0302
7 =0308 8 =0400 9 =0415 10 =0468 11 =04EA 12 =04F0 13 =04F6 14 =0502 17 =0528 18 =0533
19 =0540 20 =0546 21 =054A 24 =0584 25 =05DB 28 =05E4 30 =0601

FEATURES SUPPORTED

0NF WORD INTEGERS
1PCS

CALLFD SUPPROGRAMS

GTPSI CDRE SPM22 FSORT FAXR FADD FSUB FSURX FMPYX FDIV FLD FSTG
FSTOX FDVR FAXI IFIX FLOAT CARDZ PRNTZ SRED SREX SWRT FMPY SCOMP SFIO SIOF SIOI SUBSC
SNP SDFIO

PAGE 11

REAL CONSTANTS

.900000E 01=0076 .100000E 01=0079 .400000E 01=007A .000000E 00=007E .200000E 01=0080
.500000E 00=0082 .240000E 02=0084 .100000E-03=0086 .100000E-02=007C

INTEGER CONSTANTS

2=0088 3=0089 1=008A 0=008R 4=008C

CORE REQUIREMENTS FOR

COMMON 9770 VARIABLES 11R PROGRAM 1420

END OF COMPILATION

// XEO 1

*FILES(1,PS113)

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COLLECTION EFFICIENCY OF A CIRCULAR TUBE

GALFF = 0.500000
GARIT = 2.000000
SIGNL = -1.
DTAU = 0.020000
NIRP = 10
NSRP = 10
NX = 20

DISK STORAGE CHECK

STREAM FUNCTION FOR FLOW IN TWO CONCENTRIC PIPES WITH PARAMETERS

RA = 60.00
RR = 12.00
NP7 = 89
NPR = 61
NP7R = 65
MPRA = 13
URAT = 3.0000
UA = 1.0000
UR = 3.0000
ITERS = 2890

THE CURRENT VALUES OF PSI STORED ON DISK ARE

0.0000	0.0044	0.0177	0.0399	0.0711	0.1111	0.1599	0.2177	0.2844	0.3599	0.4444	0.5377	0.6399	0.7511	0.8711	1.0000
0.0000	0.0044	0.0178	0.0401	0.0712	0.1113	0.1602	0.2181	0.2848	0.3604	0.4448	0.5381	0.6402	0.7513	0.8712	1.0000
0.0000	0.0044	0.0178	0.0402	0.0714	0.1116	0.1606	0.2185	0.2852	0.3608	0.4452	0.5385	0.6406	0.7515	0.8713	1.0000
0.0000	0.0044	0.0179	0.0403	0.0716	0.1119	0.1610	0.2189	0.2857	0.3613	0.4457	0.5389	0.6409	0.7517	0.8714	1.0000
0.0000	0.0045	0.0180	0.0405	0.0719	0.1122	0.1614	0.2194	0.2863	0.3619	0.4462	0.5394	0.6413	0.7520	0.8716	1.0000
0.0000	0.0045	0.0181	0.0407	0.0722	0.1127	0.1620	0.2201	0.2870	0.3626	0.4469	0.5400	0.6418	0.7524	0.8719	1.0000
0.0000	0.0045	0.0182	0.0409	0.0727	0.1133	0.1627	0.2209	0.2879	0.3635	0.4478	0.5407	0.6424	0.7528	0.8720	1.0000
0.0000	0.0046	0.0184	0.0413	0.0733	0.1141	0.1637	0.2220	0.2890	0.3646	0.4488	0.5416	0.6431	0.7533	0.8723	1.0000
0.0000	0.0046	0.0186	0.0419	0.0741	0.1152	0.1650	0.2234	0.2904	0.3660	0.4501	0.5427	0.6440	0.7540	0.8726	1.0000
0.0000	0.0047	0.0190	0.0426	0.0753	0.1167	0.1667	0.2253	0.2923	0.3677	0.4516	0.5441	0.6451	0.7547	0.8730	1.0000
0.0000	0.0049	0.0196	0.0437	0.0769	0.1188	0.1690	0.2276	0.2945	0.3698	0.4535	0.5456	0.6463	0.7555	0.8734	1.0000
0.0000	0.0051	0.0205	0.0454	0.0793	0.1216	0.1721	0.2307	0.2974	0.3724	0.4557	0.5474	0.6477	0.7565	0.8739	1.0000
0.0000	0.0055	0.0219	0.0481	0.0829	0.1257	0.1762	0.2345	0.3008	0.3754	0.4582	0.5495	0.6493	0.7576	0.8744	1.0000
0.0000	0.0062	0.0243	0.0523	0.0883	0.1313	0.1815	0.2392	0.3049	0.3788	0.4611	0.5518	0.6510	0.7567	0.8750	1.0000
0.0000	0.0073	0.0284	0.0597	0.0968	0.1392	0.1891	0.2448	0.3095	0.3826	0.4642	0.5542	0.6528	0.7600	0.8757	1.0000
0.0000	0.0090	0.0355	0.0737	0.1101	0.1493	0.1958	0.2508	0.3144	0.3866	0.4674	0.5567	0.6547	0.7612	0.8763	1.0000
0.0000	0.0110	0.0456	0.1199	0.1774	0.1600	0.2036	0.2568	0.3191	0.3904	0.4705	0.5592	0.6565	0.7625	0.8767	1.0000
0.0000	0.0124	0.0512	0.1199	0.1372	0.1679	0.2099	0.2619	0.3234	0.3939	0.4733	0.5615	0.6583	0.7636	0.8770	1.0000
0.0000	0.0130	0.0527	0.1199	0.1398	0.1720	0.2141	0.2658	0.3268	0.3969	0.4759	0.5636	0.6599	0.7647	0.8781	1.0000
0.0000	0.0132	0.0531	0.1199	0.1398	0.1737	0.2165	0.2685	0.3295	0.3994	0.4781	0.5654	0.6613	0.7657	0.8786	1.0000

THE INTERVAL OF THE WRITTEN VALUES IS

THE PHYSICAL PARAMETERS ARE

REF = 0.3303983
XC = 0.794345E-01
P = 0.1374E 01

DC = 1.90500
DP = 0.0050000
RHO = 0.001213
SIGMA = 1.000000
XMU = 0.0001789
UA = 9.745A
UR = 29.237A
IS9 = 5

THE MOTION OF A CRITICAL PARTICLE IS GIVEN BY

ITER =	TAU	G(1)	G(2)	G(3)	G(4)	UZ	UR	CDRE
1	0.0000	1.002017	-0.004612	-5.000000	1.250000	1.0020	-0.0046	24.0000
0.500000	0.2000	1.002612	-0.004523	-4.799558	1.249095	1.0032	-0.0046	24.0006
1.250000	0.4000	1.003966	-0.004852	-4.598900	1.248159	1.0043	-0.0049	24.0004
2.000000	0.6000	1.005384	-0.005347	-4.397952	1.247135	1.0063	-0.0057	24.0003
	0.8000	1.007127	-0.006074	-4.196700	1.245995	1.0078	-0.0063	24.0008
	1.0000	1.009280	-0.006957	-3.995085	1.244693	1.0103	-0.0076	24.0005
	1.2000	1.01183	-0.008124	-3.793057	1.243181	1.0122	-0.0087	24.0013
	1.4000	1.014678	-0.009561	-3.590579	1.241419	1.0144	-0.0100	24.0010
	1.6000	1.017848	-0.011458	-3.387547	1.239326	1.0183	-0.0125	24.0020
	1.8000	1.021335	-0.013817	-3.183848	1.236808	1.0214	-0.0145	24.0014
	2.0000	1.025171	-0.016849	-2.979363	1.233756	1.0269	-0.0185	24.0030
	2.2000	1.029368	-0.020324	-2.773988	1.230065	1.0314	-0.0218	24.0027
	2.4000	1.033969	-0.024276	-2.567551	1.225563	1.0393	-0.0283	24.0019
	2.6000	1.039016	-0.028709	-2.359786	1.221981	1.0457	-0.0338	24.0042
	2.8000	1.044516	-0.033706	-2.150419	1.219004	1.0572	-0.0446	24.0030
	3.0000	1.050452	-0.039274	-1.939117	1.204200	1.0665	-0.0541	24.0067
	3.2000	1.056804	-0.045410	-1.725401	1.192889	1.0832	-0.0732	24.0118
	3.4000	1.063595	-0.052109	-1.508673	1.178094	1.0966	-0.0903	24.0085
	3.6000	1.119230	-0.112612	-1.288249	1.158742	1.1347	-0.1254	24.0214
	3.8000	1.150578	-0.149771	-1.061076	1.132598	1.1703	-0.1807	24.0128
	4.0000	1.17810	-0.214193	-0.827169	1.096348	1.2138	-0.2688	24.0245
	4.2000	1.275572	-0.328062	-0.581631	1.042921	1.3305	-0.4319	24.0498
	4.4000	1.445531	-0.541864	-0.313697	0.957491	1.6108	-0.7548	24.1132
	4.6000	2.195671	-0.843110	0.031483	0.810398	2.8384	-0.6267	24.4007

THE MOTION OF A CRITICAL PARTICLE IS GIVEN BY

ITER = 10
 G4LFF = 1.528320
 G4ZER = 1.529785
 G4RIT = 1.531250

TAU	G(1)	G(2)	G(3)	G(4)	UZ	UR	CDRE
0.0000	1.001737	-0.005172	-5.000000	1.529785	1.0017	-0.0051	24.0000
0.2000	1.002163	-0.005297	-4.799630	1.528744	1.0026	-0.0054	24.0005
0.4000	1.003264	-0.005708	-4.599087	1.527645	1.0036	-0.0058	24.0004
0.6000	1.004420	-0.006262	-4.398308	1.526445	1.0052	-0.0066	24.0002
0.8000	1.005826	-0.007085	-4.197280	1.525113	1.0063	-0.0074	24.0006
1.0000	1.007310	-0.008078	-3.995962	1.523597	1.0083	-0.0088	24.0004
1.2000	1.009054	-0.009383	-3.794316	1.521846	1.0099	-0.0100	24.0011
1.4000	1.011001	-0.010975	-3.592316	1.519818	1.0116	-0.0115	24.0008
1.6000	1.013358	-0.013054	-3.389887	1.517425	1.0146	-0.0141	24.0018
1.8000	1.016059	-0.015611	-3.186950	1.514567	1.0169	-0.0164	24.0012
2.0000	1.018824	-0.018397	-2.983451	1.511160	1.0209	-0.0206	24.0010
2.2000	1.022584	-0.022467	-2.779315	1.507084	1.0240	-0.0241	24.0023
2.4000	1.026567	-0.027099	-2.574399	1.502133	1.0294	-0.0306	24.0016
2.6000	1.033905	-0.033359	-2.368416	1.496097	1.0366	-0.0359	24.0040
2.8000	1.040381	-0.040775	-2.160969	1.488698	1.0448	-0.0466	24.0026
3.0000	1.048098	-0.051472	-1.952122	1.479902	1.0510	-0.0558	24.0056
3.2000	1.055963	-0.063867	-1.741715	1.467992	1.0615	-0.0736	24.0040
3.4000	1.065591	-0.081855	-1.529355	1.453473	1.0692	-0.0892	24.0087
3.6000	1.074775	-0.102978	-1.315505	1.435034	1.0808	-0.1198	24.0063
3.8000	1.091054	-0.138016	-1.099379	1.411255	1.1033	-0.1519	24.0198
4.0000	1.110081	-0.195728	-0.878923	1.379064	1.1151	-0.2116	24.0282
4.2000	1.115664	-0.248841	-0.656165	1.335886	1.1122	-0.2979	24.0180
4.4000	1.137621	-0.377618	-0.429785	1.273719	1.1353	-0.4177	24.0428
4.6000	1.128942	-0.626759	-0.201396	1.177222	1.0917	-0.7361	24.1233

THE MOTION OF A CRITICAL PARTICLE IS GIVEN BY

ITER = 20
 G4LEF = 1.529885
 G4ZER = 1.529886
 G4RIT = 1.529888

TAU	G(1)	G(2)	G(3)	G(4)	UZ	UR	CORE
0.0000	1.001737	-0.005172	-5.000000	1.529886	1.0017	-0.0051	24.0000
0.2000	1.002163	-0.005297	-4.799630	1.528845	1.0026	-0.0054	24.0005
0.4000	1.003264	-0.005708	-4.599087	1.527747	1.0036	-0.0058	24.0004
0.6000	1.004420	-0.006262	-4.398308	1.526547	1.0052	-0.0066	24.0002
0.8000	1.005826	-0.007085	-4.197280	1.525214	1.0063	-0.0074	24.0006
1.0000	1.007310	-0.008078	-3.995962	1.523699	1.0083	-0.0088	24.0004
1.2000	1.009054	-0.009383	-3.794316	1.521948	1.0099	-0.0100	24.0011
1.4000	1.011001	-0.010975	-3.592316	1.519920	1.0116	-0.0115	24.0008
1.6000	1.013138	-0.013054	-3.389887	1.517527	1.0146	-0.0141	24.0018
1.8000	1.016059	-0.015611	-3.186950	1.514669	1.0169	-0.0164	24.0012
2.0000	1.018824	-0.018397	-2.983451	1.511261	1.0209	-0.0206	24.0010
2.2000	1.022584	-0.022467	-2.779315	1.507185	1.0240	-0.0241	24.0023
2.4000	1.026567	-0.027099	-2.574399	1.502235	1.0294	-0.0306	24.0016
2.6000	1.030905	-0.033359	-2.368416	1.496198	1.0366	-0.0359	24.0040
2.8000	1.034081	-0.040775	-2.160969	1.488600	1.0448	-0.0466	24.0026
3.0000	1.034809	-0.051472	-1.952122	1.479604	1.0510	-0.0558	24.0056
3.2000	1.055963	-0.063867	-1.741715	1.468093	1.0615	-0.0736	24.0040
3.4000	1.065591	-0.081855	-1.529555	1.453574	1.0692	-0.0892	24.0087
3.6000	1.074775	-0.102978	-1.315505	1.435136	1.0808	-0.1198	24.0063
3.8000	1.098347	-0.137111	-1.099478	1.411389	1.1033	-0.1519	24.0224
4.0000	1.109863	-0.185655	-0.879219	1.379264	1.1151	-0.2116	24.0283
4.2000	1.115647	-0.248836	-0.656477	1.336091	1.1122	-0.2979	24.0180
4.4000	1.137620	-0.377618	-0.430098	1.273925	1.1353	-0.4177	24.0428
4.6000	1.125415	-0.623691	-0.202871	1.178444	1.0917	-0.7361	24.1253
4.7799	1.290820	-1.631347	0.003258	1.008198	1.6846	-1.5900	24.9538

THE UPSTREAM PARTICLE RADIUS IS 0.1529E 01

THE COLLECTION EFFICIENCY IS 0.2340E 01

THE UPSTREAM FLUID RADIUS IS 0.1730E 01

THE SAMPLING EFFICIENCY IS 0.7818E 00

// PAUS

UNCLASSIFIED

Security Classification

DOCUMENT CONTROL DATA - R & D

(Security classification of title, body of abstract and indexing annotation must be entered when the overall document is classified)

1. ORIGINATING ACTIVITY DEFENCE RESEARCH ESTABLISHMENT SUFFIELD		2a. DOCUMENT SECURITY CLASSIFICATION UNCLASSIFIED	
		2b. GROUP	
3. DOCUMENT TITLE DETERMINATION OF THEORETICAL SAMPLING EFFICIENCIES FOR ASPIRATED PARTICULATE MATTER THROUGH A DRES SAMPLING PROBE IN ANISOKINETIC FLOW (U)			
4. DESCRIPTIVE NOTES (Type of report and inclusive dates) Technical Paper			
5. AUTHOR(S) (Last name, first name, middle initial) Mellsen, Stanley B.			
6. DOCUMENT DATE June 1979		7a. TOTAL NO. OF PAGES 69	7b. NO. OF REFS 8
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13. ABSTRACT <p>Sampling efficiencies are calculated for an aspirated particulate matter sampling probe under various conditions of anisokinetic flow. A mathematical model developed for the purpose was used to obtain results for a wide range of particle sizes and flow velocities. The results can be used to predict or correct sampling errors in field or laboratory experiments. Using the same test parameters as in previous experimental tests by other workers, sampling efficiencies were calculated and the results were found to agree favorably with the results of the experiments.</p> <p style="text-align: right;">(U)</p>			

KEY WORDS

Particulate Sampling

Collection Efficiency

Sampling Probe

INSTRUCTIONS

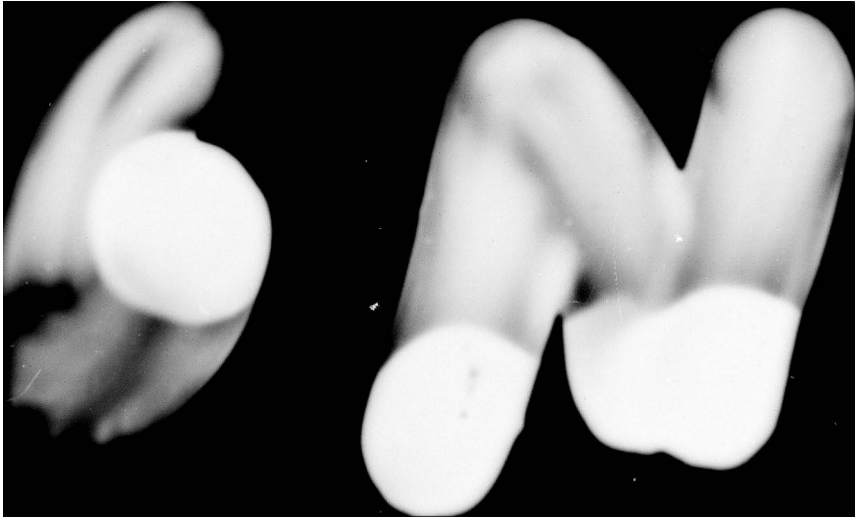
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14. **KEY WORDS:** Key words are technically meaningful terms or short phrases that characterize a document and could be helpful in cataloging the document. Key words should be selected so that no security classification is required. Identifiers, such as equipment model designation, trade name, military project code name, geographic location, may be used as key words but will be followed by an indication of technical context.

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DETERMINATION OF THEORETICAL SAMPLING EFFICIENCIES FOR
ASPIRATED PARTICULATE MATTER THROUGH A DRES
SAMPLING PROBE IN ANISOKINETIC FLOW (U)

by

Stanley B. Mellisen

ERRATA SHEET

"D" in Figures 7 to 14 denotes the diameter of the collection tube. In terms of the symbols defined under Notation this should read $2r_B$.

Figure 14 caption, second line, should read "constant flow velocity".