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**HYPERSONIC FLOW ROUGHNESS EFFECTS
EXPERIMENTAL TEST INDENTED NOSE SHAPE
HEAT TRANSFER MEASUREMENT
PRESSURE MEASUREMENT
FLOW VISUALIZATION**

20. ABSTRACT (Continue on reverse side if necessary and identify by block number)

A series of tests was performed in the Longshot Facility of the Von Karman Institute for Fluid Dynamics (VKI) in Rhode-St. Genese, Belgium, in support of USAF Grant AFOSR 78-3474. This post test report is a summary of 16 tests conducted in the Longshot Facility during the periods of 5-16 June 1978 (Phase I) and 18-29 September 1978 (Phase II). The complete test results are included in the VKI post test report. The purpose of these test was to measure local pressure and heat transfer distributions on a concave body configuration over a range of Mach number and Reynolds number conditions. Both small and large

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surface roughness body data were obtained at zero and three degrees angle of attack. High speed cine Schlieren Photographs of the model bow shock structure and dynamics were obtained.



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Hypersonic Heat Transfer Test Program
in the VKI Longshot Facility

Test Summary Report

V. DiCristina

19 January 1979

AVSD-0028-79-CR

Contract No. F49620-78-C-0029

Prepared For

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Air Force Office of Scientific Research (AFSC)
Bolling Air Force Base, D.C. 20332

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201 Lowell Street
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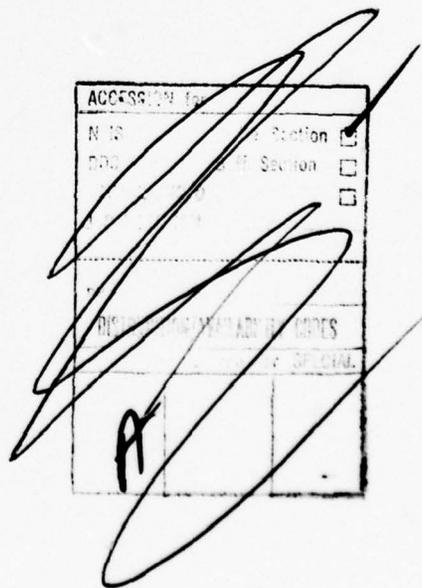
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1.0 INTRODUCTION AND SUMMARY

A series of tests were performed in the Longshot Facility of the Von Karman Institute for Fluid Dynamics (VKI) in Rhode-St. Genese, Belgium in support of USAF Great AFOSR 78-3474. This post test report is a summary of sixteen tests conducted in the Longshot Facility during the periods of 5-16 June 1978 (Phase I) and 18-29 September 1978 (Phase II). The complete test results are included in the VKI post test report.

The purpose of these tests was to measure local pressure and heat transfer distributions on a concave body configuration over a range of Mach number and Reynold number conditions. Both small and large surface roughness body data were obtained at zero and three degrees angle of attack.

The calorimeter instrumentation was calibrated at VKI during the period 24-28 April 1978. The instrumented test model was delivered to the VKI facility for the first series of tests on 5 June 1978. All data measurements were made as planned including high speed cine schlieren photographs of the model bow shock structure and dynamics.

The results of the first eight tests are included here, the complete data set including the large roughness body data will be issued as part of the final VKI test report.

2.0 DISCUSSION

2.1 TEST FACILITY

The VKI Longshot facility was used for this program. Longshot differs from a conventional gun tunnel in that a heavy piston is used to compress the nitrogen test gas to very high pressure and temperatures. The test gas is then trapped in a reservoir at peak conditions by the closing of a system of check valves. The flow conditions decay monotonically during 20 milliseconds running time as the nitrogen trapped in the reservoir flows through the 6° half angle conical nozzle into the pre-evacuated open jet test chamber. The maximum supply conditions used in these tests are approximately 60,000 lb/in² at 1900°K to 2350°K. These conditions provide unit Reynolds numbers of 8.5×10^6 per foot at a Mach number of 16 and 3.5×10^6 at $M = 20$.

2.2 MODEL DESIGN

The test model design was a concave body configuration shown in Figure 1. This model configuration was designated as model M. The model was fabricated from 303 stainless steel having a wall thickness of approximately 0.2 inches. The first series of tests (runs 1-8) were performed with a small 5 mil roughness over the model surface. The rough surface was achieved by sand blasting. The second series of tests (runs 9-16) were performed with large 65 mil roughness elements over the model surface. The large roughness was created by bonding aluminum spheroid particles onto a sprayed adhesive and coated with a sprayed acrylic. The local calorimeter and pressure port areas did not contain roughness elements.

2.3 INSTRUMENTATION

The test model was instrumental with both pressure and heat flux gages. The gage locations are shown in Figure 1. The test model contains eleven heat flux gages and nine pressure taps. Only eight pressure measurement channels were available at the test facility. The last pressure tap on the concave surface was not connected. PCB model 112A21 high resolution pressure transducers were connected through short flexible tubes to the surface taps.

The heat transfer gages are smooth calorimeter discs fabricated from .004 inch high conductivity copper stock. A 1 mil wire chromel alumel thermocouple is welded to the copper disc which is bonded to an insulating holder. Twenty calorimeters were fabricated and calibrated at the VKI facility. Table I lists the measured calibration constant (C_T) for each gage. The values shown are the coverage of two measurements. The actual measured disc thickness (X_A) is also shown compared with the calculated effective disc thickness (X_E). The data was derived from the following relationships:

$$C_T = \frac{\dot{Q}_C}{\Delta E / \Delta t}$$

$$X_E = \frac{C_T}{\rho C_P} = .00523 C_T$$

where: \dot{Q}_C = calibration heat flux
 $\Delta E / \Delta t$ = thermocouple EMF output
 X_E = Effective gage thickness
 C_T = Calibration constant
 ρ = Density, copper
 C_P = Specific heat, copper

The variations in X_E are due to the thermocouple weld joints.

2.4 TEST MATRIX

The matrix for the sixteen runs made in the Phase I and II test series is shown in Table II. The small surface roughness model is designated as MR1 and the large roughness model as MR2. The pressure and heat transfer gages were intermixed on the model surface as shown in Figure 1. In order to obtain a complete distribution of pressure and heat transfer measurements for the angle of attack cases both a positive and negative incidence angle was run for the same test conditions as shown in Table II.

2.5 TEST RESULTS

The data measurements for each run included surface pressure, heat transfer and schlieren photos of the model flow field.

The model flow field and bow shock structure was recorded using a 6000 spark per second cine schlieren system. Figures 2 and 3 show the recorded bow shock time sequence. As expected the model contour generates an unstable flow field which results in an oscillating bow shock. The observed oscillations indicate a frequency of 1500-2000 cps. The Strouhal number which depends on the flow frequency (f), model diameter (d), and freestream velocity (V_∞) is:

$$St = Df/V_\infty$$

From the observed oscillations the average Strouhal number appears to be $St = 0.13$ which is about 30% lower than previously observed from tests at $M_\infty = 10$ and $Re_\infty = 8 \times 10^6$ ft^{-1} .

The measured pressure distributions are shown in Figures 4 to 11. The pressure data has been normalized with respect to the measured pitot pressure measurement. The heat transfer distributions are shown in Figures 12 to 19. The heat transfer data are the measured values for each run condition.

TABLE I

Heat Transfer Gage Calibration Data

<u>Gage No.</u>	<u>Avg. Calib. Coeff., C_T</u>	<u>Effect. Gage Thickness, X_E (in.)</u>	<u>Actual Gage Thickness (in.)</u>
1	.733	.0038	.0031
2	.671	.0035	.0036
3	.530	.0028	.0035
4	.555	.0029	.0038
5	.654	.0034	.0038
6	.548	.0029	.0032
7	.480	.0025	.0034
8	.456	.0024	.0036
9	.495	.0026	.0034
10	.667	.0035	.0038
11	.783	.0041	.0037
12	.951	.0050	.0035
13	.580	.0030	.0031
14	.606	.0032	.0034
15	.612	.0032	.0034
16	.626	.0033	.0037
17	.595	.0031	.0036
18	.585	.0031	.0035
19	.505	.0026	.0036
20	.676	.0035	.0033

TABLE II

VKI Longshot Test Matrix

Run No.	Model	(degree)	M_{∞}	Re_{∞} (ft ⁻¹)
<u>Phase I</u>				
1	MR1	0	20	2.0×10^6
2	MR1	0	20	3.3×10^6
3	MR1	0	16	4.5×10^6
4	MR1	0	16	8.6×10^6
5	MR1	+3	20	3.3×10^6
6	MR1	-3	20	3.3×10^6
7	MR1	+3	16	6.6×10^6
8	MR1	-3	16	8.6×10^6
<u>Phase II</u>				
9	MR2	0	20	2.0×10^6
10	MR2	0	20	3.3×10^6
11	MR2	0	16	4.5×10^6
12	MR2	0	16	8.6×10^6
13	MR2	+3	20	3.3×10^6
14	MR2	-3	20	3.3×10^6
15	MR2	+3	16	8.6×10^6
16	MR2	-3	16	8.6×10^6

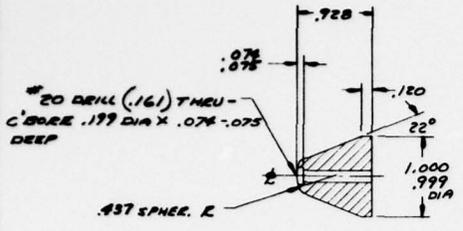
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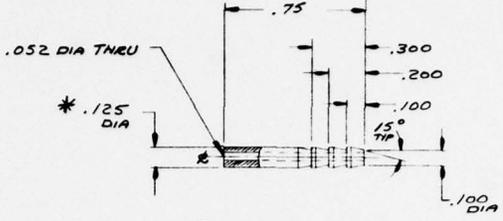
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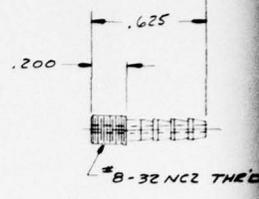
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② BLUNT NOSE - 1 REQ'D
 MAT'L - EXISTING (DWG T990-630-P2)
 NOTE: PLUG EXISTING PRESSURE HOLE (.052 DIA)

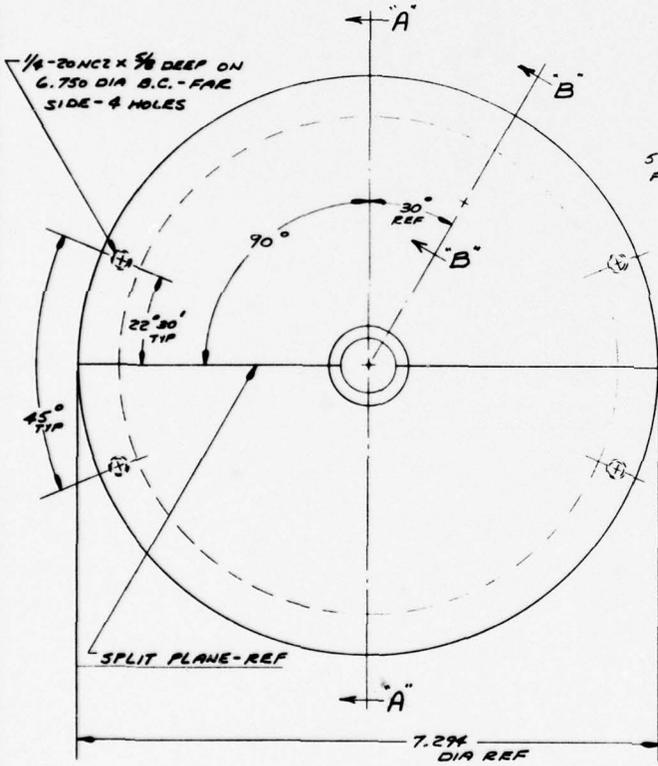


③ PRESSURE TUBE - 5 REQ'D
 MAT'L - BRASS
 NOTE: DIA MARKED * TO BE PRESS FIT WITH PART #1



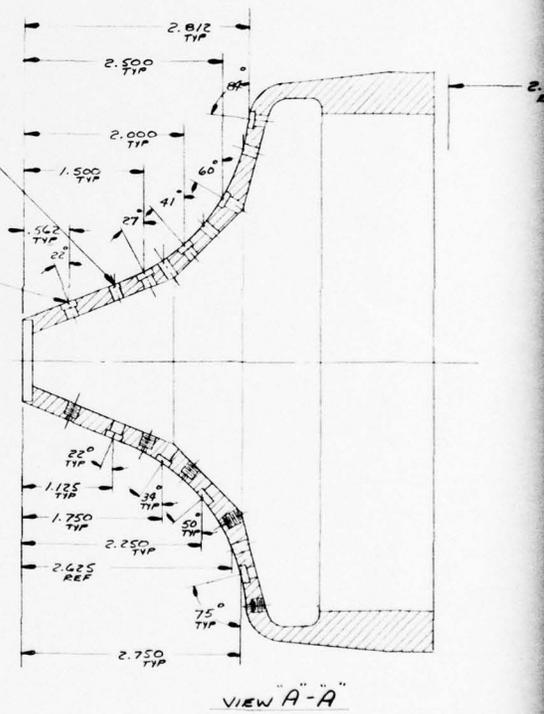
④ PRESSURE TUBE -
 MAT'L - #8-32 N.C.Z. X 1.0
 OTHERWISE SAME AS PART #1

C



① MODEL BODY - 1 REQ'D
 MAT'L - MAKE FROM DWG T990-630-PART #1
 NOTE: PLUG, LEAK TIGHT, ALL EXISTING HOLES EXCEPT AS SHOWN
 PLUGS TO BE SMOOTH & FLUSH WITH OUTSIDE CONTOUR

.125 DIA THRU -
 5 HOLES TOTAL (PRESS FIT WITH PART #3)
 .161 DIA THRU - C BORE
 .199 DIA X .078 - .075 DEEP -
 10 HOLES TOTAL



VIEW "A-A"

A

PART CLASS:		DASH NO.	
DWG CLASS:		NEXT ASSY	
ASG BY:			

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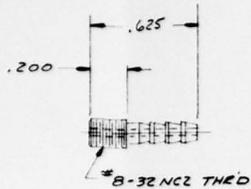
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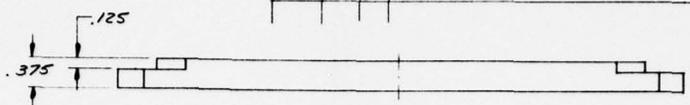
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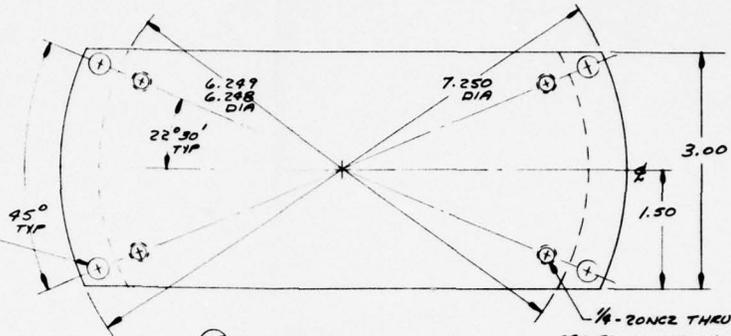
ENGR	REVISIONS					
	CHG	ZONE	LTR	DESCRIPTION	DATE	APPROVED
		A		GEN'L CHGS	4-7-78	JRC



④ PRESSURE TUBE-THREADED - 5 REQ'D
 MAT'L - *B-32 UNC X 1.00 LG BRASS SCREW
 OTHERWISE SAME AS PART #3

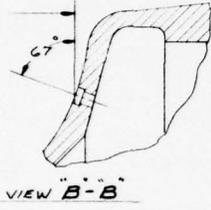


.281 DIA THRU - 4 HOLES
 SPACED AS SHOWN ON
 6.750 DIA B.C.

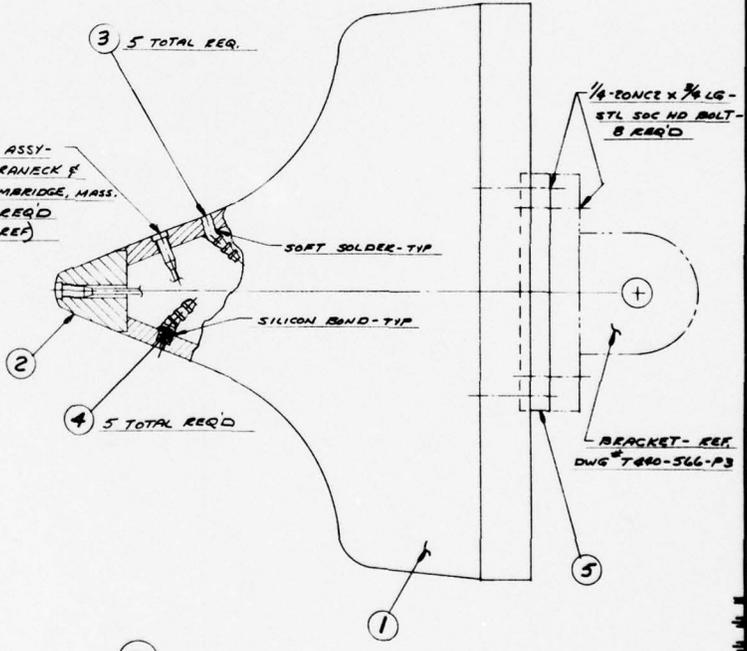


⑤ SUPPORT PLATE - 1 REQ'D
 MAT'L - STN STL 303

1/4 - 20 UNC THRU - 4 HOLES
 SPACED AS SHOWN ON
 5.625 DIA B.C.



HEAT FLUX GAGE ASSY -
 VENDOR: BOLT, BERANECK &
 NEWMAN CO. - CAMBRIDGE, MASS.
 11 TOTAL REQ'D
 (DWG T490-691-P3, REF)



⑥ ASSY

NOTE: BEND PRESSURE TUBES APPROX 30° TOWARD REAR @ 2 FORWARD
 LOCATIONS FOR CLEARANCE @ ASSY

QTY. REQ'D	CODE IDENT	PART OR IDENTIFYING NO.	NOMENCLATURE OR DESCRIPTION	MATERIAL OR NOTE	SPECIFICATION	ZONE	ITEM NO.

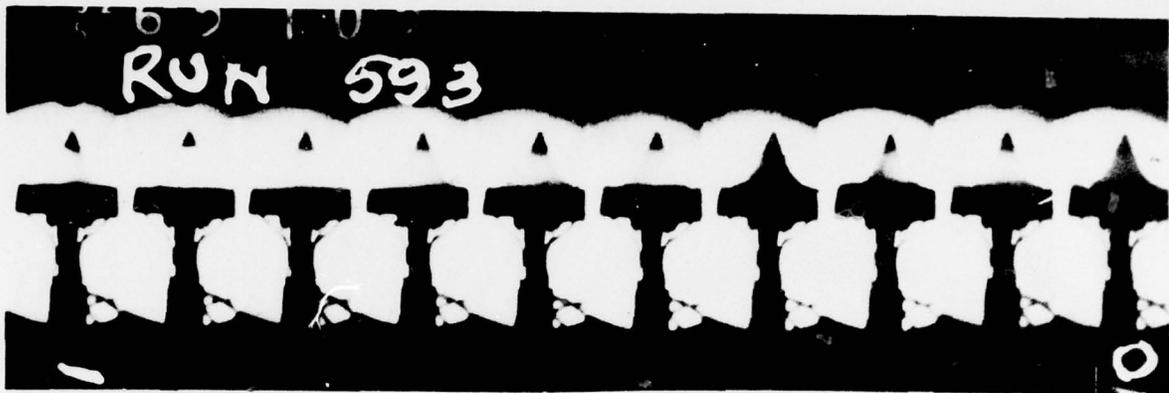
LIST OF PARTS

UNLESS OTHERWISE SPECIFIED - TOLERANCE AND DRAWING INTERPRETATION PER D143 X ± .08 .XX ± .04 .XXX ± .020 MACHINED ANGLES - 90° 30' SHEET METAL BEND ANGLES - 2° P10001 SURFACE ROUGHNESS ✓		CONTR. NO. RELEASE	 WILMINGTON, MASSACHUSETTS 01827	
DRAWN: JRC CHECKED: JRC DATE: 3-17-78 DESIGN APPROVAL		TITLE LONGSHOT TEST MODEL CONFIGURATION "M" V.K.I.		
MANAGER RECT CH DESIGNER		SIZE D	CODE IDENT NO. 04614	DWG NO. T490-747
DASH NO.		SCALE FULL		INSET

DASH NO.	NEXT ASSY	USED ON	QTY. PER NHA	END ITEM NUMBER	SERIAL NO.	APPLICATION	EFFECTIVITY

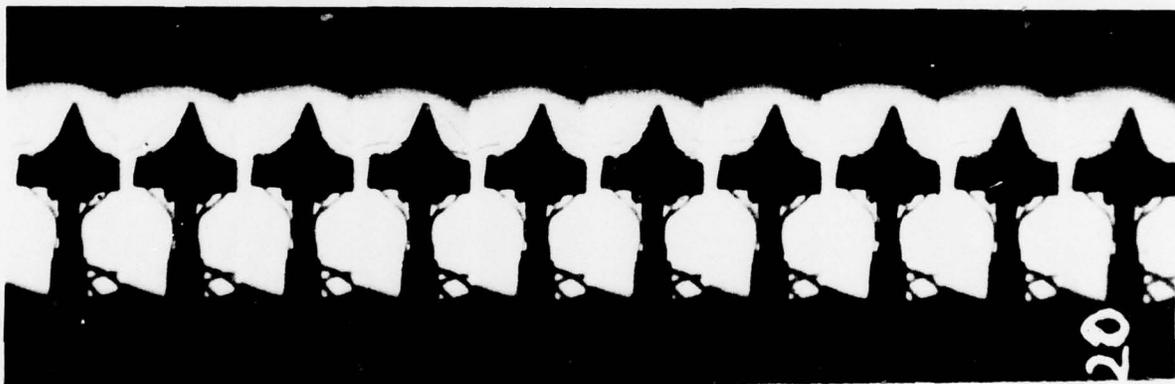
FIGURE 1

FIGURE 2a
Ciné Schlieren Flow Field
Run No. 593 Model MR1
 $M_\infty = 16$, $Re_\infty = 8.6 \times 10^6/\text{ft}$, $\alpha = -3^\circ$



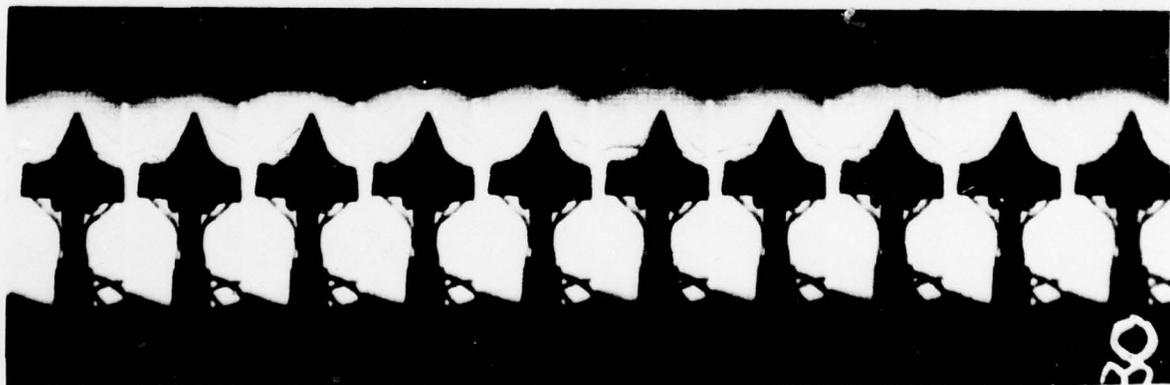
Frame 1

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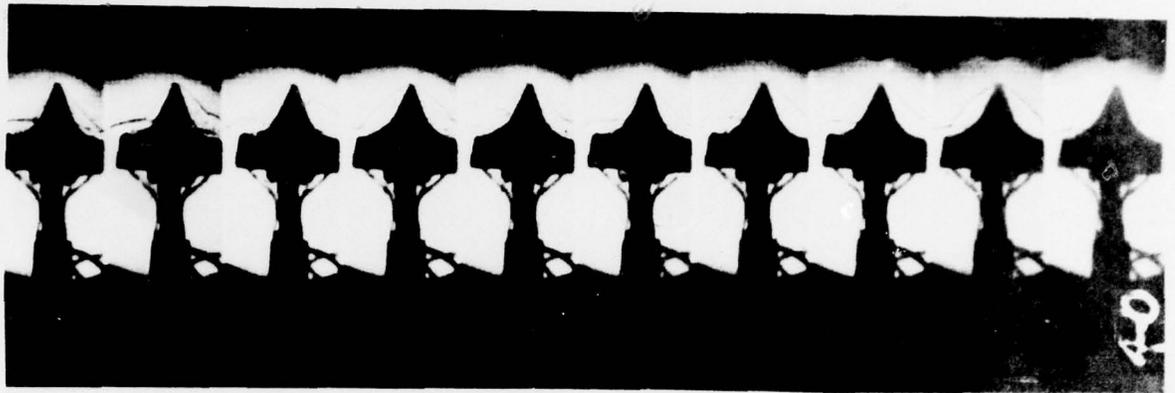
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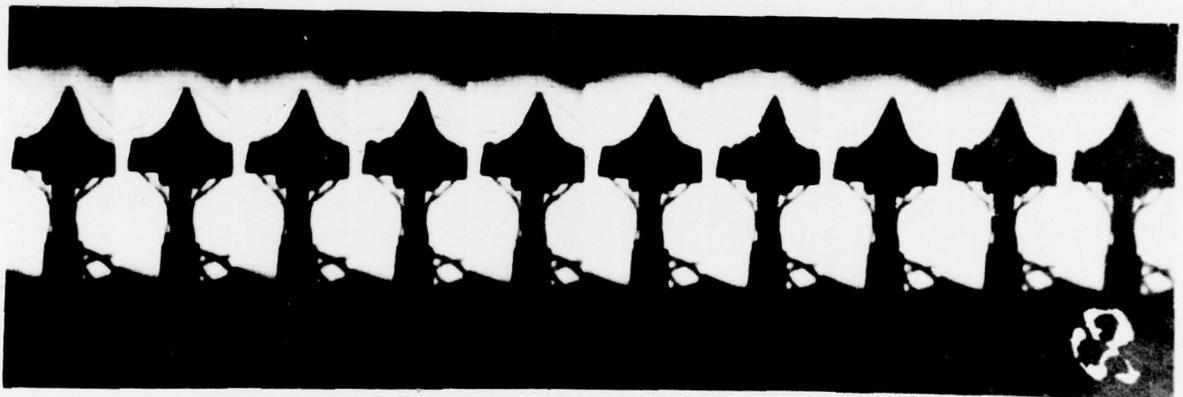
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Figure 2b
Ciné Schlieren Flow Field
Run No. 593 Model MR1
 $M_\infty = 16$, $Re_\phi = 8.6 \times 10^6/\text{ft}$, $\alpha = -3^\circ$



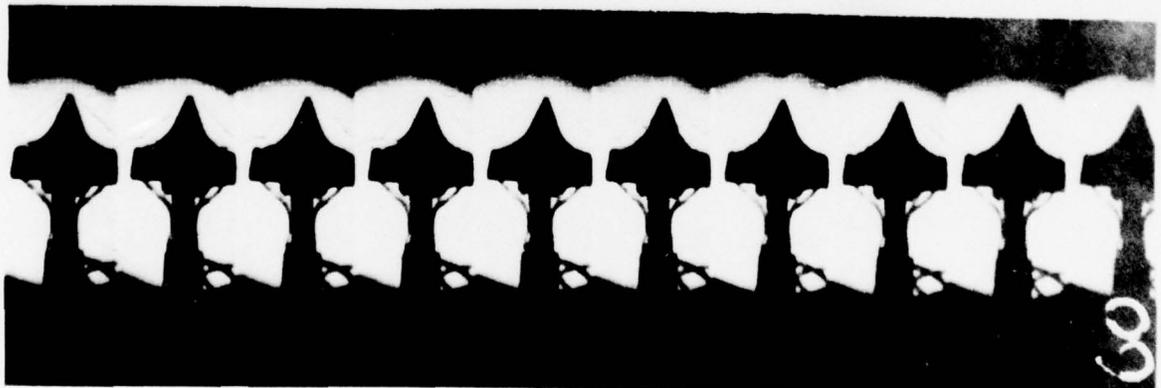
Frame 31

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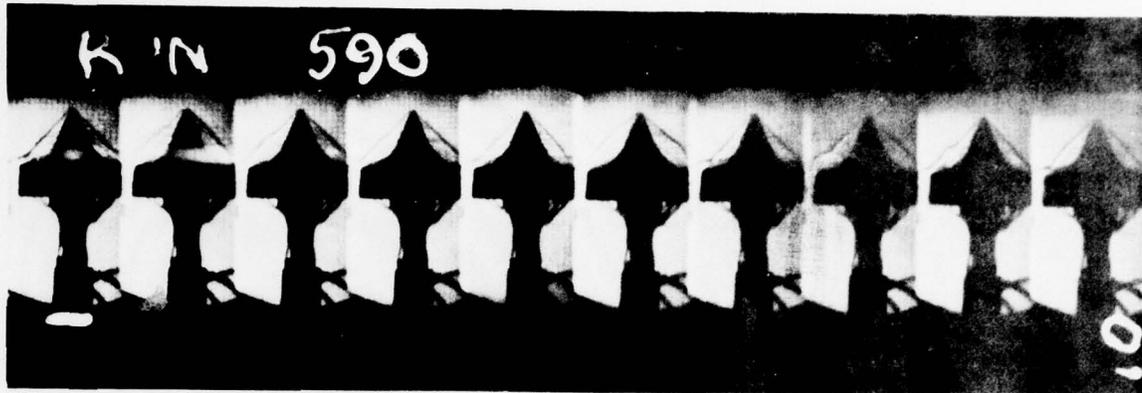
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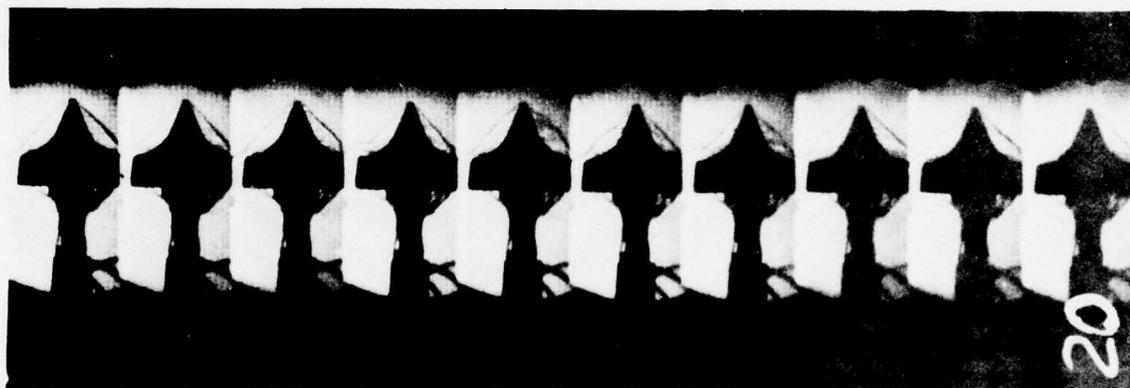
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Figure 3a
 Cine Schlieren Flow Field
 Run No. 590 Model MRL
 $M_\infty = 20$, $Re_\infty = 2.0 \times 10^6/\text{ft}$, $\alpha = 0^\circ$



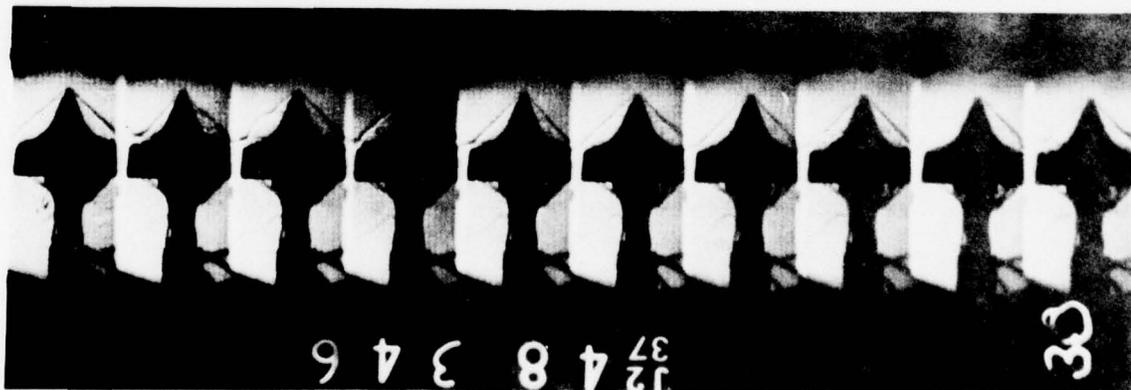
Frame 1

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11

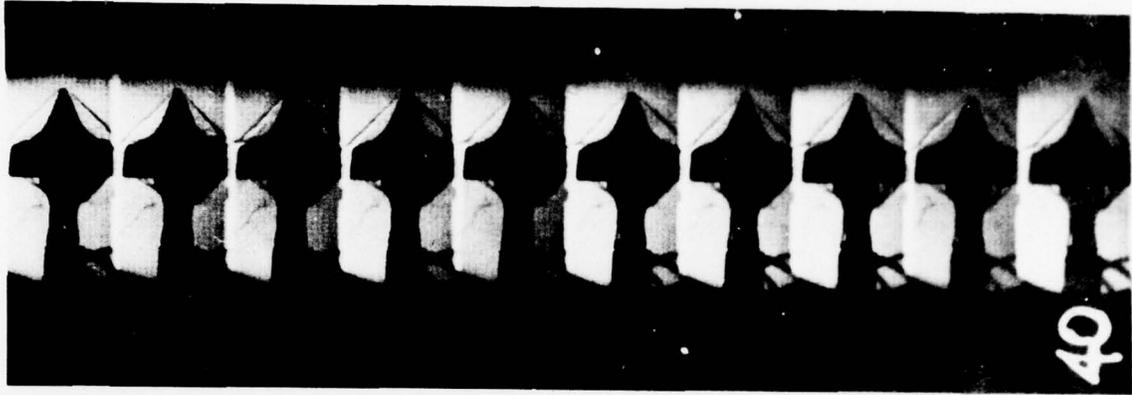
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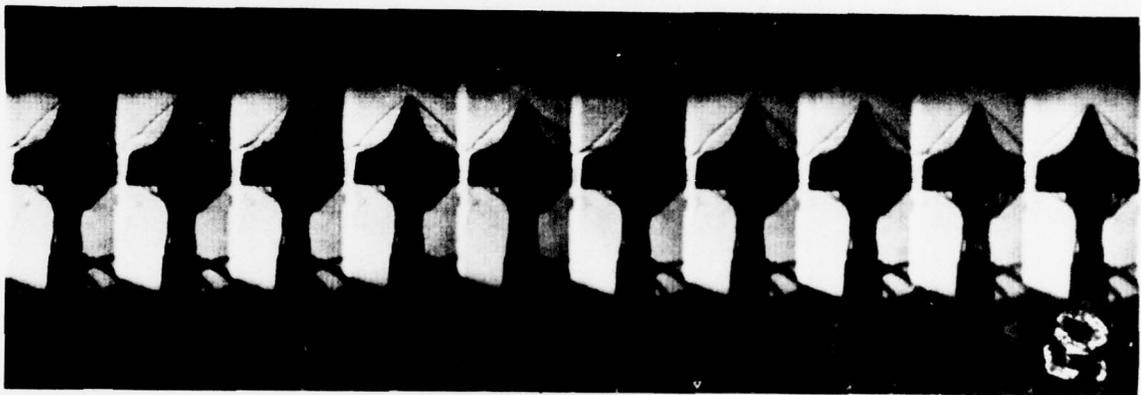
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Figure 3b
Cine Schlieren Flow Field
Run No. 590 Model MR1
 $M_\infty = 20, Re_\infty = 2.0 \times 10^6/\text{ft}, \alpha = 0^\circ$



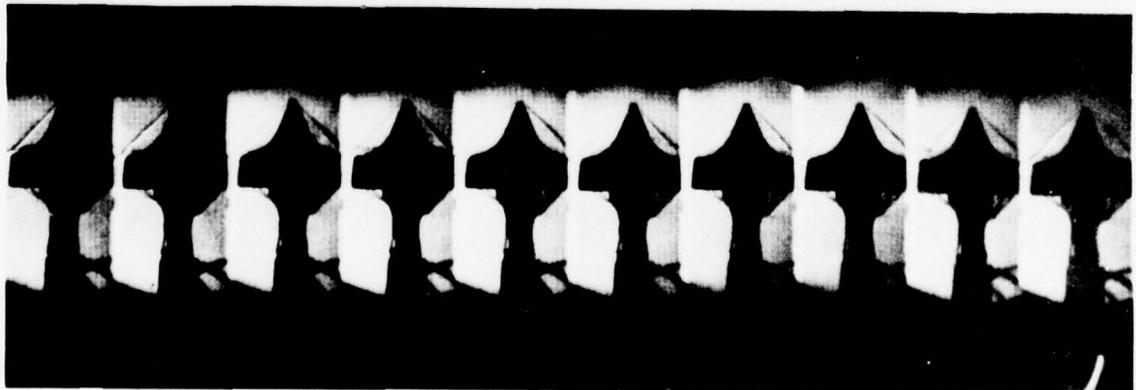
Frame 31

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41

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51

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FIGURE NO. 16

HEAT TRANSFER ON CONCAVE BICONIC

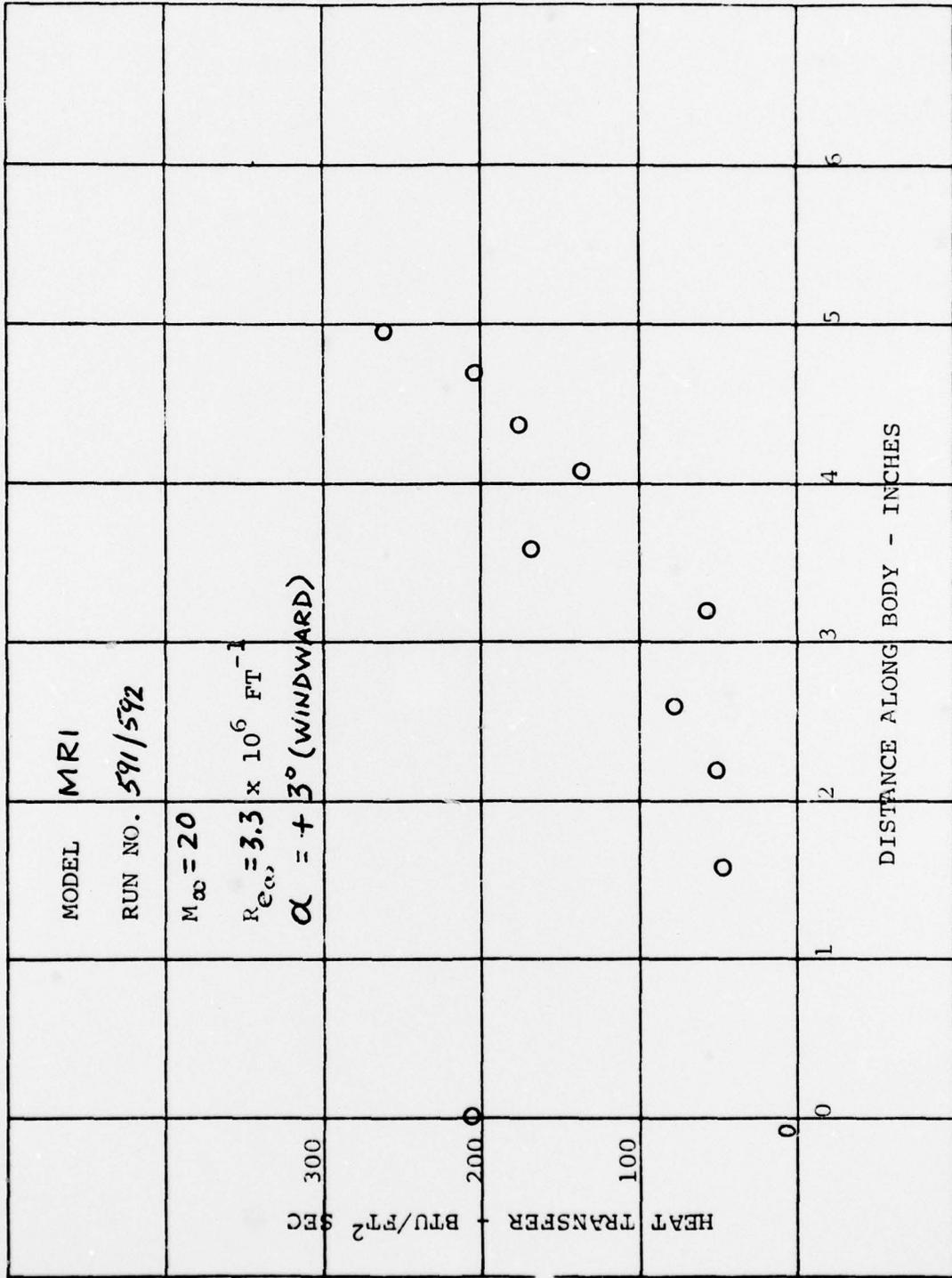


FIGURE NO. 18

HEAT TRANSFER ON CONCAVE BICONIC

