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# OPTIMIZATION OF SURFACE CONTOURS FOR ELASTIC BODIES IN CONTACT

by T. S. ARORA and J. S. ARORA

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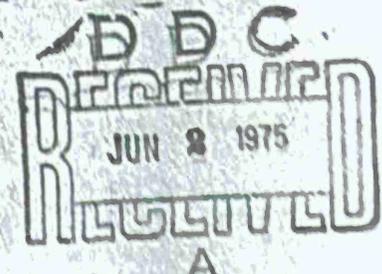
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FOR ELASTIC BODIES IN CONTACT

by

T. S. Arora and J. S. Arora

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TABLE OF CONTENTS

	Page
LIST OF FIGURES . . . . .	iv
LIST OF TABLES . . . . .	vi
LIST OF SYMBOLS . . . . .	vii
 Chapter	
I. INTRODUCTION . . . . .	1
II. THE UNBONDED CONTACT PROBLEM FOR ELASTIC BODIES IN CONTACT . . . . .	4
2.1 Analytical Formulation of the Problem . . . . .	4
2.2 Sequential Linear Programming Solution Technique . . . . .	7
III. CONTACT SURFACE DESIGN PROBLEMS FOR BEAMS ON ELASTIC FOUNDATIONS . . . . .	14
3.1 Beam on Elastic Foundation with No Initial Gap . . . . .	16
3.2 Initially Bent Beam on an Elastic Foundation . . . . .	29
IV. CONCLUSIONS . . . . .	50
Suggestions . . . . .	51
BIBLIOGRAPHY. . . . .	52
APPENDICES . . . . .	55
APPENDIX A: FORMULATION OF MATRICES B, A, a, and C . . . . .	56
APPENDIX B: PROGRAM LISTING . . . . .	62

## LIST OF FIGURES

Figure		Page
1.	Geometry of Contacting Bodies. . . . .	3
2.	Potential Contact Points for the Beam on Elastic Foundation. . . . .	15
3.	Contact Stress vs. Contact Length from Quadratic Programming Solution. . . . .	17
4.	Contact Stress vs. Contact Length with $U = 0.01$ . . . . .	22
5.	Gap Size vs. Potential Contact Point with $U = 0.01$ . . . . .	23
6.	Contact Stress vs. Contact Length. . . . .	24
7.	Gap Size vs. Potential Contact Point. . . . .	25
8.	Contact Stress vs. Contact Length with Augmented Cost Function. . . . .	26
9.	Gap Size vs. Potential Contact Point with Augmented Cost Function. . . . .	27
10.	Contact Stress vs. Contact Length from Quadratic Programming Solution with $G = 0.0001$ . . . . .	31
11.	Contact Stress vs. Contact Length with $U = 0.01$ and $G = 0.0001$ . . . . .	33
12.	Gap Size vs. Potential Contact Point with $U = 0.01$ and $G = 0.0001$ . . . . .	34
13.	Contact Stress vs. Contact Length with $G = 0.0001$ . . . . .	35
14.	Gap Size vs. Potential Contact Point with $G = 0.0001$ . . . . .	36

Figure	Page
15. Contact Stress vs. Contact Length with Augmented Cost Function and with $G = 0.0001$ . . . . .	37
16. Gap Size vs. Potential Contact Point with Augmented Cost Function and with $G = 0.0001$ . . . . .	38
17. Contact Stress vs. Contact Length from Quadratic Programming Solution with $G = 0.001$ . . . . .	42
18. Contact Stress vs. Contact Length from Quadratic Programming Solution with $G = 0.005$ . . . . .	43
19. Contact Stress vs. Contact Length with $G = 0.001$ . . . . .	44
20. Gap Size vs. Potential Contact Point with $G = 0.001$ . . . . .	45
21. Contact Stress vs. Contact Length with $G = 0.005$ . . . . .	46
22. Gap Size vs. Potential Contact Point with $G = 0.005$ . . . . .	47
23. Two Bodies in Contact. . . . .	58

LIST OF TABLES

TABLE	Page
1. COMPARISON OF RESULTS FOR LOADS $t = 1000$ lbs. AND $t = 2000$ lbs. FOR BEAM ON ELASTIC FOUNDATION . . . . .	28
2. CONTACT STRESS DISTRIBUTION AND FINAL GAP FOR BEAM ON ELASTIC FOUNDATION $t = 1000$ lbs., $U = 0.01$ . . . . .	30
3. COMPARISON OF RESULTS FOR LOADS $t = 1000$ lbs. AND $t = 2000$ lbs. FOR AN INITIALLY BENT BEAM ON ELASTIC FOUNDATION . . . . .	39
4. CONTACT STRESS DISTRIBUTION AND FINAL GAP FOR INITIALLY BENT BEAM ON ELASTIC FOUNDATION $t = 1000$ lbs., $U = 0.01$ . . . . .	40
5. COMPARISON OF RESULTS FOR BEAM ON ELASTIC FOUNDATION WITH NO INITIAL GAP AND WITH INITIAL GAPS FOR $U = 0.01$ . . . . .	48
6. COMPARISON OF RESULTS FOR BEAM ON ELASTIC FOUNDATION WITH NO INITIAL GAP AND WITH DIFFERENT INITIAL GAPS FOR $t = 2000$ lbs. . . . .	49

## LIST OF SYMBOLS

### Symbol

B	Matrix formed from influence coefficients of the two bodies.
A	Affine transformation accounting for rigid body displacement of Body 1.
a	Constant vector that depends on the externally applied loads to the two bodies and the initial gap between them at the potential contact region.
b	The vector of contour modifications.
C	The external load column vector for externally applied loads.
$\epsilon$	Final gap size vector.
t	Total applied load.
w	Work done.
S	Contact stress vector for potential contact points.
J	Cost function.
$b_{n+1}$	Upper bound on contact stresses
P	Projection matrix = $\begin{bmatrix} 0 & \vdots & I_{n-m} \end{bmatrix}$
$I_{n-m}$	An identity matrix of dimension (n-m).
$t^1$	External force vector on Body 1.
$\hat{I}$	Index set for points in contact.
$\tilde{I}$	Index set for points not in contact.
NC	Number of points in contact

Symbol

NNC	Number of points not in contact.
$b^0$	Lower bound on contour modification.
$b^1$	Upper bound on contour modification.
H	Matrix that gives rigid body displacement at the points of application of $t^1$ .
$X_i$	X-coordinate of ith point.
q	Rigid body coordinate vector for Body 1.
$d_i$	Initial gap between ith potential contact points.
n	Number of potential contact points.
m	Number of degrees of freedom.

Chapter I

INTRODUCTION

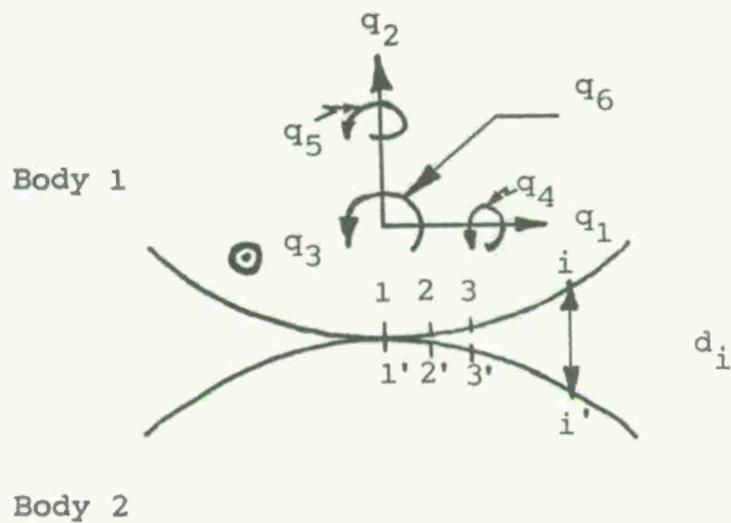
Many problems in continuum mechanics and mechanical system design involve elastic bodies that come into contact with each other, under applied loads. Such problems, called contact problems of elasticity, are non-classical in the sense that one does not initially know the contacting region. Considerable research has been pursued in recent years to develop constructive methods of determining the contact region and contact stress distribution [1-4].

The geometry of two bodies that will come into contact under applied load is shown schematically in Figure 1(a). A numerical method of solution, by quadratic programming [5] has been developed, in which one selects a set of potential contact points, as indicated in Figure 1(a). The selection of potential contact points is explained in detail in [6]. The initial gap between the  $i$ th potential contact points is denoted by  $d_i$ . The rigid body displacement coordinates of Body One are indicated in Figure 1(a) and are represented by a generalized coordinate vector  $q$ , of dimension one to six. Once the potential contact points are defined on the two

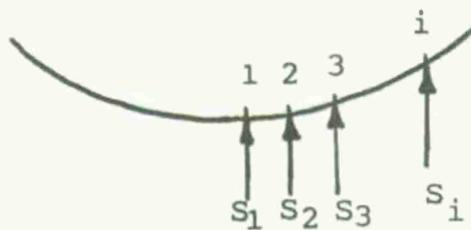
bodies, contact stress may be replaced by equivalent point loads, or contact forces, denoted as  $S_i$  in Figure 1(b).

When the bodies come into contact, a force distribution over the surfaces arises and high contact stresses may occur over some parts of the contact region. This is undesirable, since plastic deformation of the bodies may occur, or high normal forces may lead to wear of machine parts that move relative to each other. This can be quite important in precision machines. The objective of this work is to develop a technique for adjusting the contour of one or both of the bodies in order to achieve a minimum peak contact stress between the bodies, under a given load.

A related problem was solved by Conry [6], in which the design objective was to select the contour to achieve a constant stress over the contact arc. A linear programming method was presented in [7,8] to implement this design objective. In general, one cannot achieve a constant stress distribution over the contact arc when kinematic constraints define limits on modification of the contours of the bodies. This will be particularly critical in precision cams and fasteners that require precise location of parts. The design objective selected in this work is to minimize the peak contact stress, subject to constraints on the extent of modification of surface contours.



1(a).



1(b).

Figure 1. Geometry of Contacting Bodies.

## Chapter II

THE UNBONDED CONTACT PROBLEM FOR  
ELASTIC BODIES IN CONTACT2.1 Analytical Formulation of the Problem

The elastic contact problem formulation of [6] will be employed as the model of the elastic contact problem. Only a brief summary of this analytical formulation will be presented here. The gap variable  $\mathcal{E}$ , due to a stress distribution  $S$ , is given by the equation

$$\mathcal{E} = BS + Aq + a + b \quad (2.1.1)$$

where  $B$  is a matrix formed from influence coefficients of the two bodies,  $A$  is an affine transformation accounting for rigid body displacement of Body 1,  $a$  is a constant vector that depends on the externally applied loads to the two bodies and the initial gap between them at the potential contact region, and  $b$  is the vector of contour modification to be selected. Formulae for  $B$ ,  $A$ , and  $a$  are given in Appendix A.

Equilibrium of Body 1 is obtained through direct application of the principle of virtual work, which results in the linear equation

$$A^T S = C \quad (2.1.2)$$

where the column vector  $C$  depends on the externally applied load, and is given in Appendix A.

Compatibility conditions between the two contacting bodies require that the product of each gap variable and contact stress be zero. Analytically, this is

$$\epsilon_i S_i = 0, \quad i = 1, 2, \dots, n \quad (2.1.3)$$

where  $n$  is the number of potential contact points. This condition may be interpreted as stating that either the gap or the contact stress must be zero at each point on the contacting bodies. Further, the gap and contact stress must be non-negative. Analytically these conditions are

$$S_i \geq 0, \quad \epsilon_i \geq 0, \quad i = 1, 2, \dots, n \quad (2.1.4)$$

As is shown in [6], Equations (2.1.1)-(2.1.4) are the Kuhn-Tucker necessary conditions for solution of a convex quadratic programming problem, which may be stated as

$$\begin{array}{l} \text{minimize} \quad A^T S + \frac{1}{2} S^T B S \\ \text{subject to constraints} \quad \left. \begin{array}{l} A^T S = C \\ S \geq 0 \end{array} \right\} \end{array} \quad (2.1.5)$$

This problem may be solved through direct application of the simplex technique of quadratic programming [5]. It is also shown in [5] that if a solution exists, it is unique and may be reliably determined, through application of a simplex technique for quadratic programming [6].

The cost function to be minimized in this design problem is the maximum contact stress,

$$J = \text{Max}_i S_i \quad (2.1.6)$$

Since this maximum value function is difficult to treat analytically, it may be replaced by defining an upper bound  $b_{n+1}$  on the contact stress, through the set of inequalities

$$\phi_i = S_i - b_{n+1} \leq 0 \quad (2.1.7)$$

The cost function  $J$  of equation (2.1.6) may now be replaced by an equivalent problem of minimizing the upper bound on contact stress

$$\bar{J} = b_{n+1} \quad (2.1.8)$$

subject to constraints (2.1.7).

Finally, it is often required that both upper and lower bounds be placed on modifications of the surface contours of the two bodies. Analytically, these conditions may be stated as

$$\phi_{n+i} = b^0 - b_i \leq 0, \quad i = 1, \dots, n \quad (2.1.9)$$

and

$$\phi_{2n+i} = b_i - b^1 \leq 0, \quad i = 1, \dots, n \quad (2.1.10)$$

where  $b^0$  and  $b^1$  are the lower and upper bounds on contour modification, respectively.

The design problem may now be stated as follows: Determine the design variable vector  $b$  and the upper bound  $b_{n+1}$  to minimize  $\bar{J}$ , subject to constraints (2.1.1) - (2.1.4), (2.1.7), (2.1.9) and (2.1.10).

## 2.2 Sequential Linear Programming Solution Technique

With an initial estimate of the contour (normally  $b = 0$ ), one may solve the contact-analysis problem of equations (2.1.1) - (2.1.4) to obtain the contact arc and contact stresses. Using this solution, define the index sets

$$\hat{I} = \{i_1, i_2, \dots, i_{NC}\}$$

of points in contact and

$$\tilde{I} = \{j_1, j_2, \dots, j_{NNC}\}$$

of points not in contact, where NNC is the number of points not in contact. The stresses at points in contact will be denoted as the NC dimensional vector  $\hat{S}$ .

Define  $\hat{B}$  as the matrix B with rows and columns corresponding to points not in contact removed,  $\hat{A}$  as the matrix A with rows corresponding to the points not in contact removed, and the vectors  $\hat{a}$  and  $\hat{b}$  as the vectors a and b with components corresponding to points not in contact removed.

One may now state the set of conditions that must be satisfied if the contact arc is not changed; namely

$$\hat{E} \equiv \hat{B}\hat{S} + \hat{A}q + \hat{a} + \hat{b} = 0 \quad (2.2.1)$$

The equilibrium equations (2.1.2) must still be satisfied with contact loads applied only on the fixed contact region, so one obtains

$$\hat{A}^T \hat{S} = C \quad (2.2.2)$$

The objective now is to determine the reduced design variable  $\hat{b}$  so that peak contact stresses are reduced

as much as possible, keeping the same contact region and satisfying conditions (2.1.7) - 2.1.10).

The gap variable outside the prescribed contact region must remain non-negative, so

$$\tilde{\epsilon} \equiv \tilde{B}\hat{S} + \tilde{A}q + \tilde{a} + \tilde{b} \geq 0 \quad (2.2.3)$$

where  $\tilde{B}$  is the matrix  $B$  with columns corresponding to points not in contact and rows corresponding to the points in contact removed,  $\tilde{A}$  is the matrix  $A$  with rows corresponding to points in contact removed, and vectors  $\tilde{a}$  and  $\tilde{b}$  are the vectors  $a$  and  $b$  with rows corresponding to points in contact removed. Here, the reduced vector  $\tilde{b}$  represents the estimates of the contour design variable from the preceding iteration. Finally, it is necessary to impose the condition,

$$\hat{S} \geq 0 \quad (2.2.4)$$

The design modification problem is now to determine the reduced vector  $\hat{b}$  and  $b_{n+1}$  to minimize

$$\bar{J} = b_{n+1}$$

subject to: (2.2.5)

$$\begin{aligned}
 \hat{S} &\leq b_n + 1 && (2.2.5, \text{con'd.}) \\
 b^0 - \hat{b} &\leq 0 \\
 \hat{b} - b^1 &\leq 0 \\
 \hat{B}\hat{S} + \hat{A}q + \hat{a} + \hat{b} &= 0 \\
 \hat{A}^T \hat{S} &= c \\
 \tilde{\mathcal{E}} = \tilde{B}\hat{S} + \tilde{A}q + \tilde{a} + \tilde{b} &\geq 0 \\
 \hat{S} &\geq 0
 \end{aligned}$$

This formulation for determining the modified contour variable is simply a linear programming problem in the variables  $\hat{S}$ ,  $q$ , and  $\hat{b}$  which can be solved with standard linear programming codes.

Following each solution of the linear programming problem, the contact stresses and gap variables must be evaluated to determine the contact surface to be employed in the next iteration. Any potential contact point for which  $\tilde{\mathcal{E}}_j = 0$  is included in the contact surface for the next iteration. At any points in the preceding potential contact surface for which  $\hat{S}_j = 0$ , it is presumed that separation will occur in the subsequent iteration, so these points are deleted from the vectors  $\hat{\mathcal{E}}$  and  $\hat{S}$ . These modification rules form the basis for definition of the next linear programming problem to be solved for the optimum contour on an adjusted contact arc. This process is continued until no new points

come into contact and no points that were previously in contact are released.

Each linear programming solution will reduce the value of the peak stress, until the process stops. Analytically, this condition can be written as follows

$$\left. \begin{array}{l} b_{n+1}^{(0)} \geq b_{n+1}^{(1)} \\ b_{n+1}^{(1)} \geq b_{n+1}^{(2)} \\ \vdots \\ b_{n+1}^{(k-1)} \geq b_{n+1}^{(k)} \end{array} \right\} \quad (2.2.6)$$

where the superscript on  $b_{n+1}$  denotes the iteration number. Combining (2.2.6) one obtains

$$b_{n+1}^{(0)} \geq b_{n+1}^{(1)} \geq \dots \geq b_{n+1}^{(k)} \quad (2.2.7)$$

This gives a sequence of non-increasing real numbers, called a monotone sequence, that is bounded below by zero. Any non-increasing sequence that is bounded below is convergent [9], so the sequence of solutions of linear programming problems (2.2.5) must converge.

It was observed from preliminary calculations that as the value of  $U$  is increased, peak stress decreases, but a stage is reached when stress at all the points in

the contact region is the same. The peak stress obtained in this case is a local minimum and no contour design variable reaches its allowable limit. This situation can be avoided by adding a penalty function to the cost function that is intended to broaden the contact region. The penalty function used here is as follows

$$J = b_{n+1} + \delta_1 \sum_{j=1}^{NNC} \tilde{\epsilon}_j \quad (2.2.8)$$

where  $\delta_1$  is a small constant greater than zero. The linear programming with the augmented cost function (2.2.8) finds an absolute minimum of the peak contact stress.

The process described above is summarized in the following algorithm:

- Step 1. Estimate the design variables (normally  $b^{(0)} = 0$ ).
- Step 2. Solve for  $\hat{S}$  and  $\mathcal{E}$  by quadratic programming, using the necessary conditions (2.1.1)-(2.1.4).
- Step 3. Form index sets  $\hat{I}$  and  $\tilde{I}$  for points of contact ( $S_i \geq 0, \mathcal{E}_i = 0$ ) and for points not in contact ( $\mathcal{E}_j \geq 0, S_j = 0$ ), respectively.
- Step 4. Construct  $\hat{B}, \hat{A}, \hat{a}$ , and  $\hat{b}$  for  $i \in \hat{I}$  and  $A, B, a$ , and  $b$  for  $j \in \tilde{I}$ .

- Step 5. Solve the linear programming problem (2.2.5) for  $\hat{S}$ ,  $q$ , and  $\hat{b}$ . (The cost function may be changed as given in (2.2.8) if necessary).
- Step 6. Evaluate  $\tilde{\mathcal{E}}$  from (2.2.3).
- Step 7. From new  $\hat{I}$  to include points for which  $\mathcal{E}_j = 0$  and deleting points for which  $S_j = 0$  and form  $\tilde{I}$  of points for which  $\mathcal{E}_j \geq 0$  and  $S_j = 0$ .
- Step 8. If  $\hat{I}$  is unchanged, terminate; otherwise return to Step 4.

## Chapter III

CONTACT SURFACE DESIGN PROBLEMS FOR  
BEAMS ON ELASTIC FOUNDATIONS

Examples are presented to demonstrate the technique developed in Chapter II. Two cases of the design problem with a beam on an elastic foundation are considered: (1) with no initial gap; and (2) with an initial gap.

A point load is applied to the center of the beam, as shown in Figure 2(b). Only vertical displacement of the center of the beam is chosen to represent its rigid body displacement. Formulation of the matrices B, A, a, b, and C for this problem is explained in detail in [2]. A quadratic programming problem is solved to obtain the contact arc as input to the linear programming problem.

The design variables  $b_i$  are limited by the constraint  $b_i \leq U \left(1 - \frac{x_i^2}{L^2}\right)$ , where  $x_i$  is the horizontal distance of the  $i$ th potential contact point from the center of the beam,  $L$  is half the potential contact length (as only half of the beam is considered due to symmetry), and  $U$  is a constant greater than zero.

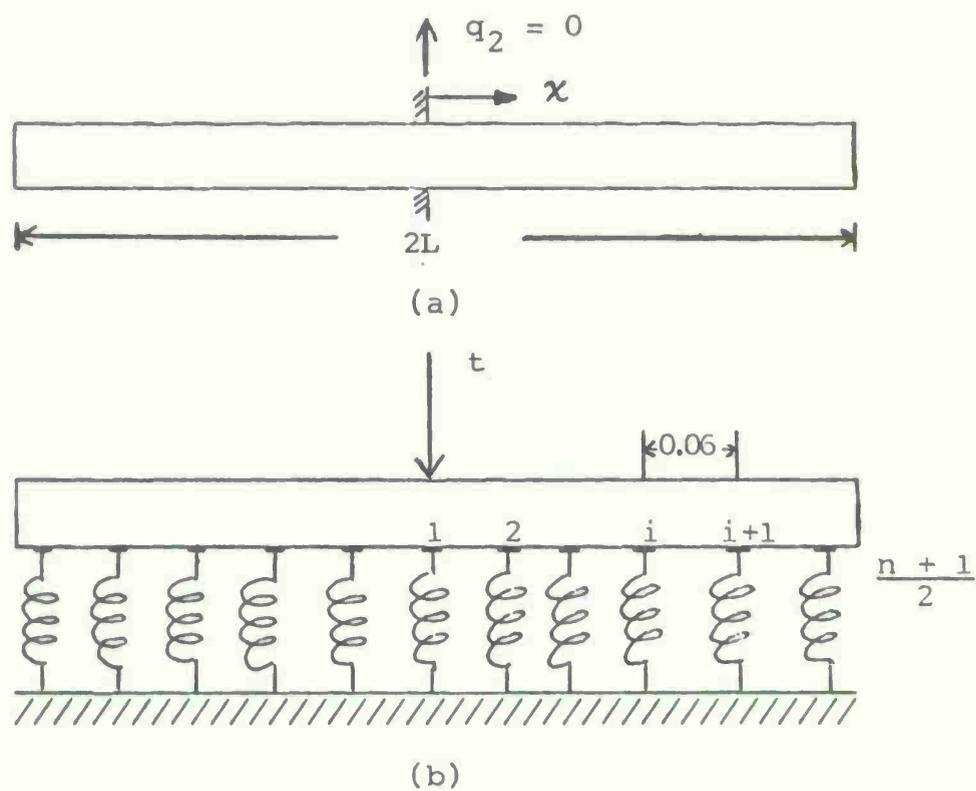


Figure 2. Potential Contact Points for the Beam on Elastic Foundation.

### 3.1 Beam on Elastic Foundation with No Initial Gap

The following material properties are used: Young's modulus of the beam material = 340,000 psi; diagonal element of the flexibility matrix for the elastic foundation = 0.00005 in./lb.; height of the beam = 0.5 in.; and width of the beam = 1.0 in.

The quadratic programming problem was solved for 25 potential contact points on half of the beam, with an interval size of 0.06 in. Potential contact length, as shown in Figure 3(a) is taken as 2.88 in. The contact length obtained numerically was 1.38 in., with loads of 1000 lbs. and 2000 lbs. A quadratic programming code called ZORILLA [10] was used to solve this quadratic programming problem. Results are presented in Figure 3(b).

At this stage, one has the contact arc to be employed in the linear programming problem of (2.2.5). Equations (2.2.5) can now be written as:

$$\begin{aligned}
 \bar{J} &= b_{n+1} \\
 \hat{S} - b_{n+1} &\leq 0 \\
 \hat{b} &\geq b^0 \\
 \hat{b} &\leq b^1 \\
 \hat{B}\hat{S} + \hat{A}q + \hat{b} &= -\hat{a} \\
 \hat{A}^T \hat{S} &= c \\
 \tilde{B}\hat{S} + \tilde{A}q &\geq -\tilde{a}-\tilde{b} \\
 \hat{S} &\geq 0
 \end{aligned} \tag{3.1.1}$$

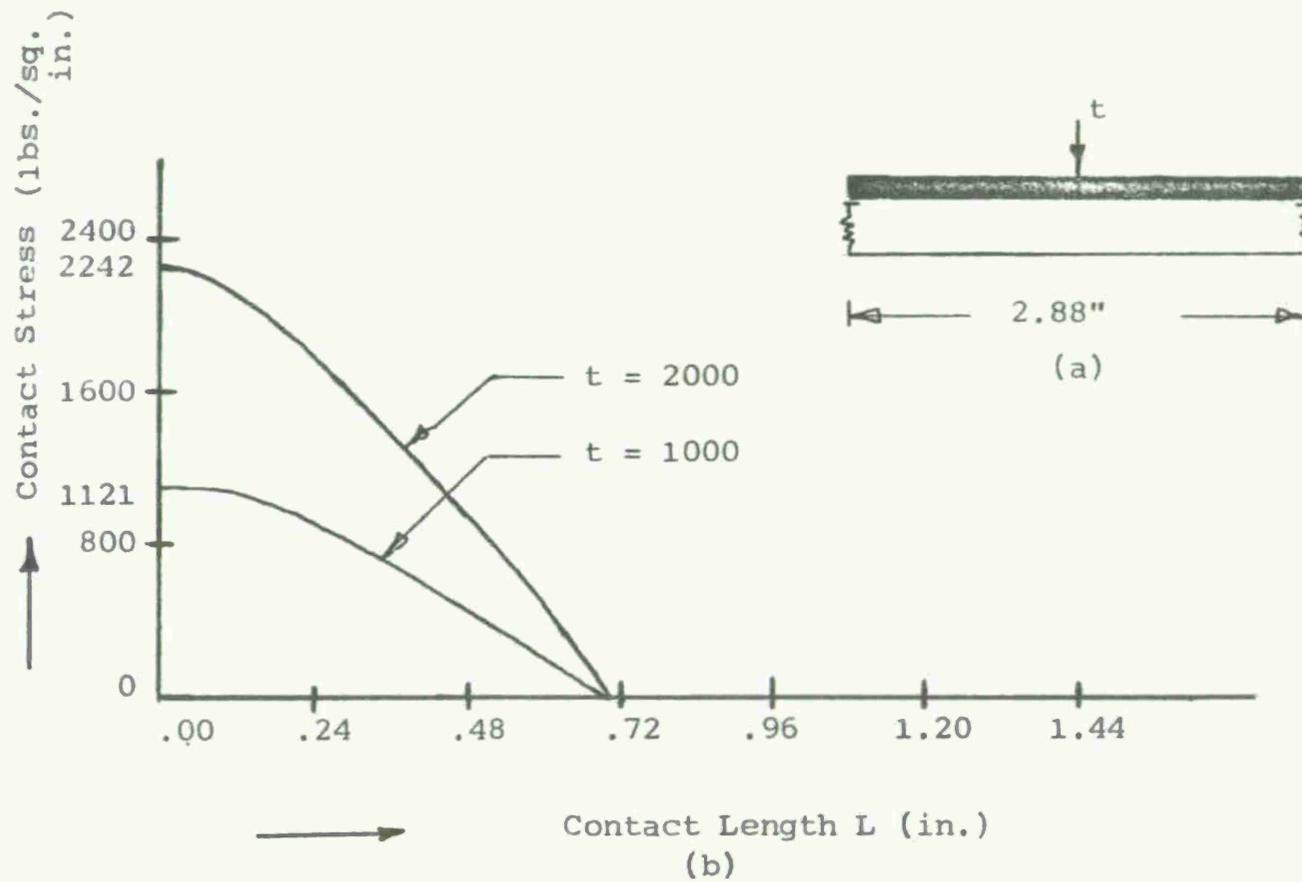


Figure 3. Contact Stress vs. Contact Length from Quadratic Programming Solution.

The linear programming subroutine used to solve (3.1.1) is explained in [11] and is summarized in Appendix B. This subroutine requires that the right hand side values of constraints must be given as a separate vector, all elements of which must be positive. If  $b^0 = -U[1 - \frac{x^2}{L^2}]$  and  $b^1 = U[1 - \frac{x^2}{L^2}]$ , then Equations (3.1.1) can be re-written as

$$\left. \begin{aligned}
 \bar{J} &= b_{n+1} \\
 \hat{S} - b_{n+1} &\leq 0 \\
 -\hat{b} &\leq -b^0 \\
 \hat{b} &\leq b^1 \\
 \hat{B}\hat{S} + \hat{A}q + \hat{b} &= 0, \text{ as } \hat{a} = 0 \\
 \hat{A}^T \hat{S} &= C \\
 -\tilde{B}\hat{S} - \tilde{A}q &\leq \tilde{a} + \tilde{b}
 \end{aligned} \right\} \quad (3.1.2)$$

The linear programming subroutine used solves only maximization problems, so the cost function is replaced by  $J = -b_{n+1}$ . Since  $q_r$  can be positive or negative, the vector  $q$  is decomposed into positive and negative parts as follows:

$$\begin{aligned}
 q &= q^+ - q^- \\
 q^+ &= q && \text{when } q \geq 0 \\
 &= 0 && q \leq 0 \\
 q^- &= q && \text{when } q \leq 0 \\
 &= 0 && q \geq 0
 \end{aligned} \quad (3.1.3)$$



$$\bar{B} \equiv \begin{array}{c} \text{NC} \\ \text{NC} \\ \text{NC} \\ \text{NC} \\ \text{m} \\ \text{NNC} \end{array} \begin{bmatrix} \text{m} & \text{m} & 1 & \text{NC} & \text{NC} \\ \hline 0 & 0 & -I & 0 & I \\ \hline 0 & 0 & 0 & I & 0 \\ \hline 0 & 0 & 0 & -I & 0 \\ \hline \hat{A} & -\hat{A} & 0 & I & \hat{B} \\ \hline 0 & 0 & 0 & 0 & \hat{A} \\ \hline -\tilde{A} & \tilde{A} & 0 & 0 & \tilde{B} \end{bmatrix}$$

contains constraint coefficients. Its dimension is  $(4 * \text{NC} + \text{m} + \text{NNC}, 2 * \text{m} + 1 + 2 * \text{NC} + \text{NGE})$  and the vector  $RQ^T = [ 0 \mid -b^0 \mid b^1 \mid 0 \mid c \mid \tilde{a} + \tilde{b} ]$ , of dimension  $(1, 4 * \text{NC} + \text{m} + \text{NNC})$ , is the vector of right hand side values of the constraints.

The design problem was solved with loads  $t = 1000$  lbs. and  $t = 2000$  lbs., with a range of values of contour

modification limit  $U$ . Results are presented in Figures (4 - 9). All the calculations were done on an IBM 360/65 computer. Table 1 shows the comparison of results for different loads and values of  $U$  ranging from 0.01 to 0.05. It is observed that the contact length for the optimum design, with a fixed value of  $U$ , varies with applied load. However, when the value of  $U$  is changed in the same proportion as the loads, it is noted that the contact length is the same for both cases (and peak contact stress are proportional). As the value of  $U$  is increased, the contact length appears to increase and the value of peak stress decreases. A stage is reached when stress at each point in the contact region is the same, which stops the iterative process. In the case of  $U = 0.025$ ,  $t = 1000$  lbs., and  $U = 0.05$ ,  $t = 2000$  lbs., no contour design variable's constraint reaches its tolerance limit. The cost function was augmented with penalty function and the linear programming problem was solved for the above case. The results are presented in Figures (8 - 9). It is also observed that the contour design variable constraint at the center point of the contact region is always tight.

Table 1 presents peak stress, computing time and contact lengths for several numerical examples. Solutions of the linear programming problem converge in at most 8

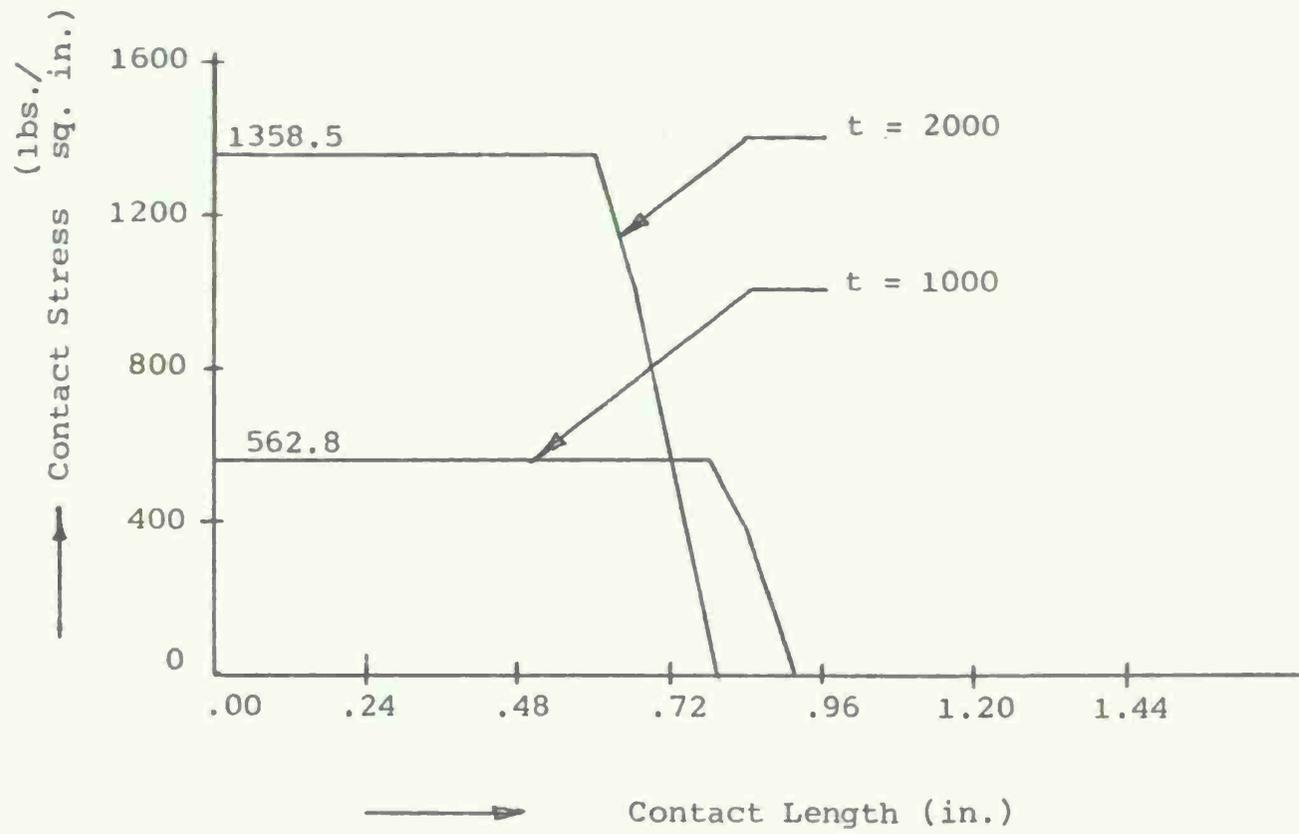


Figure 4. Contact Stress vs. Contact Length with  $U = 0.01$ .

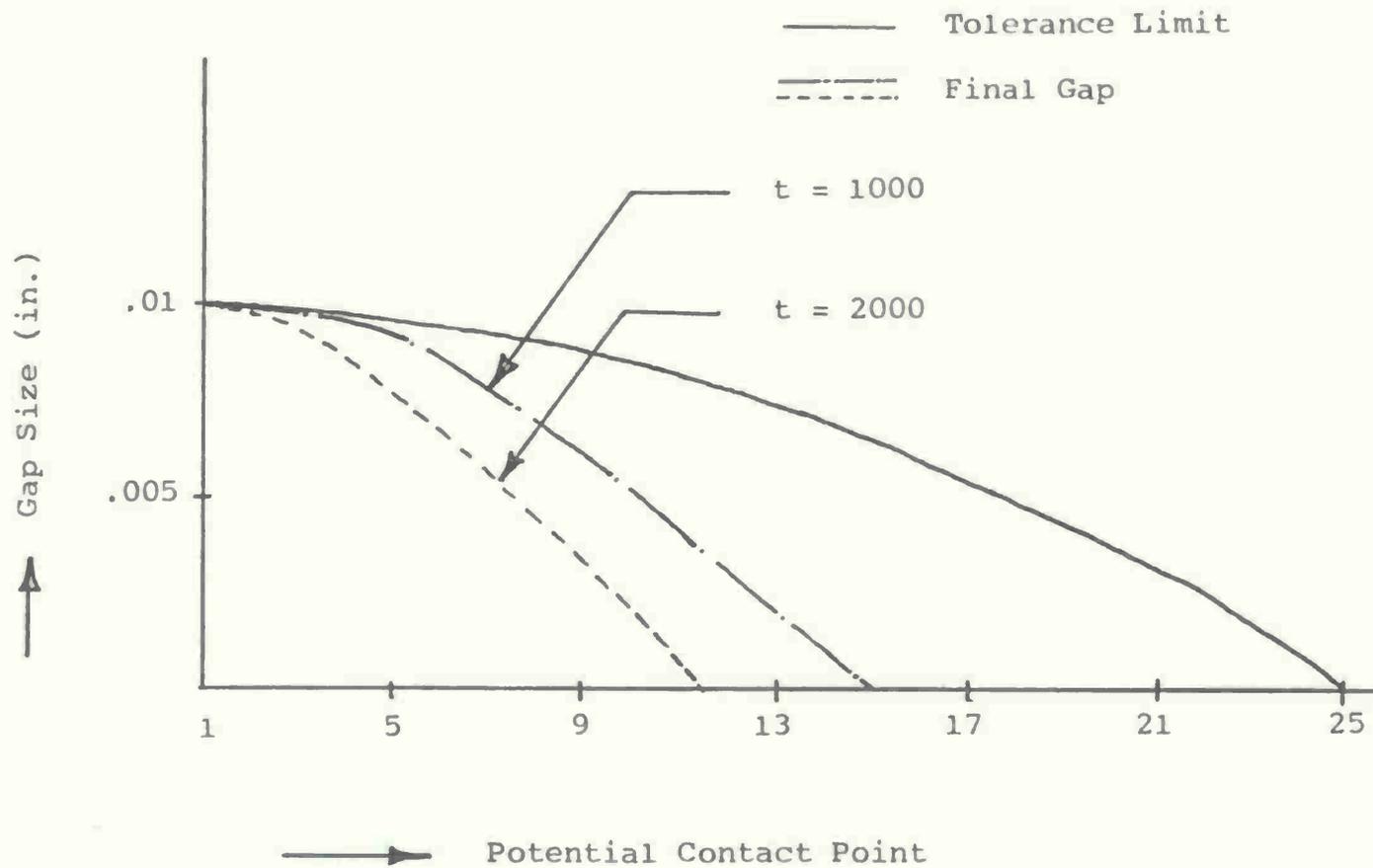


Figure 5. Gap Size vs. Potential Contact Point with  $U = 0.01$ .

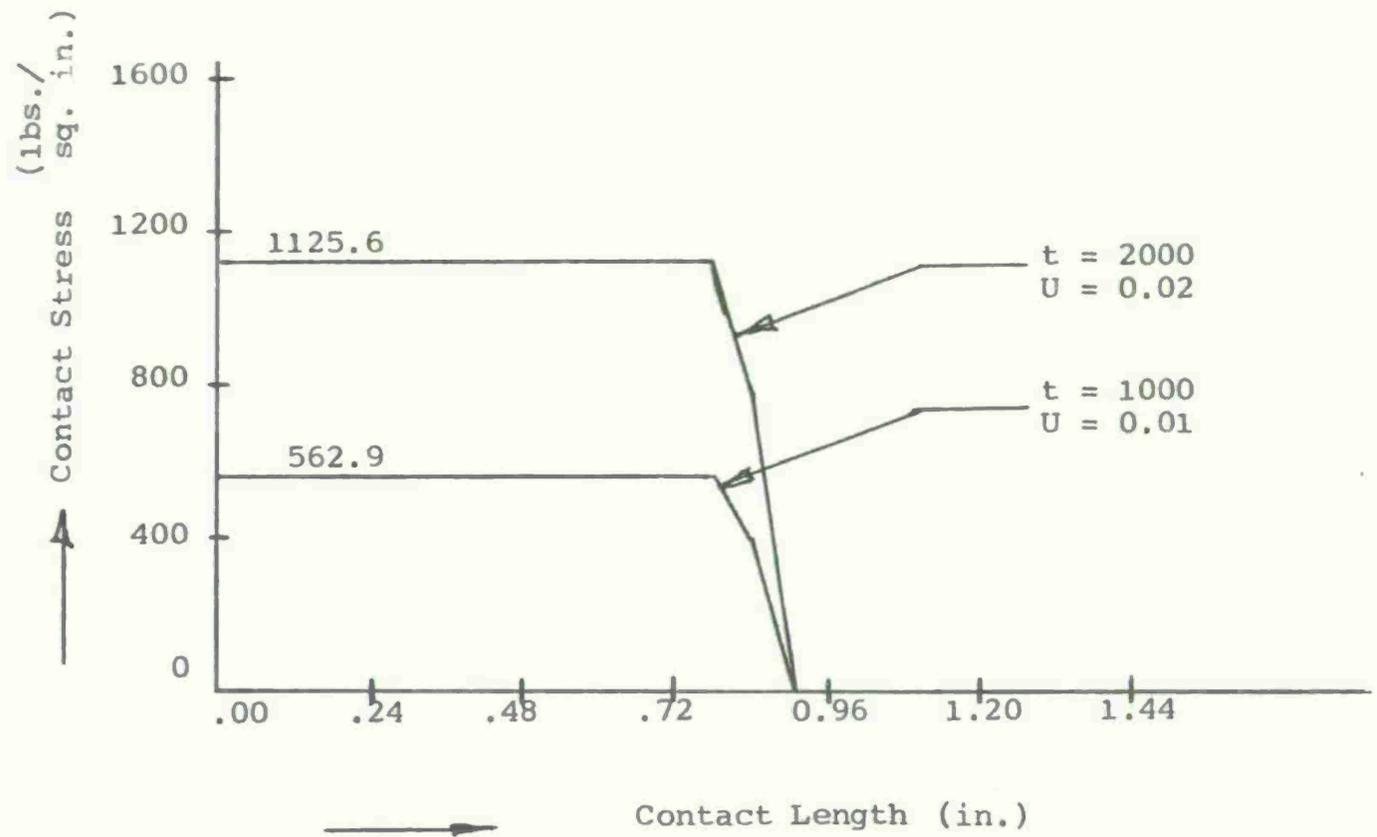


Figure 6. Contact Stress vs. Contact Length.

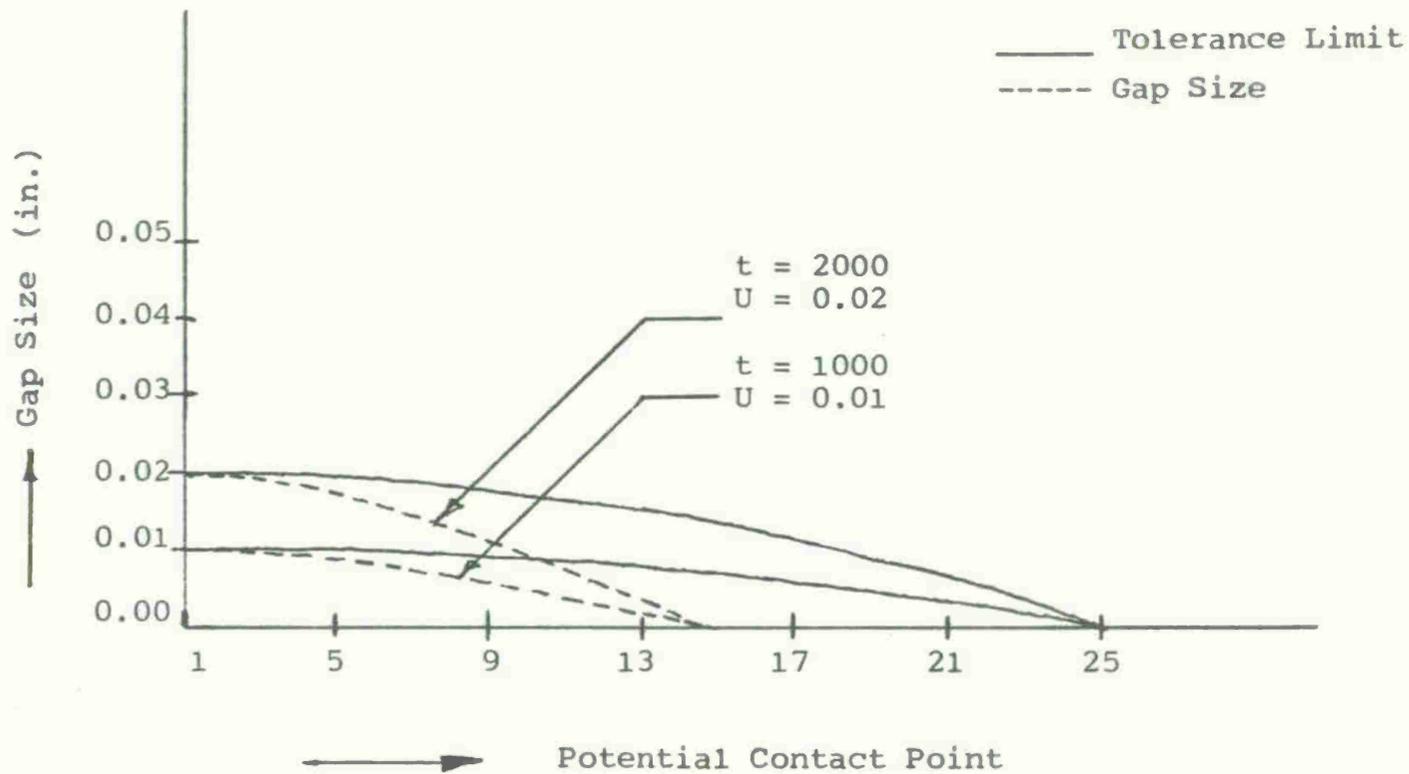


Figure 7. Gap Size vs. Potential Contact Point.

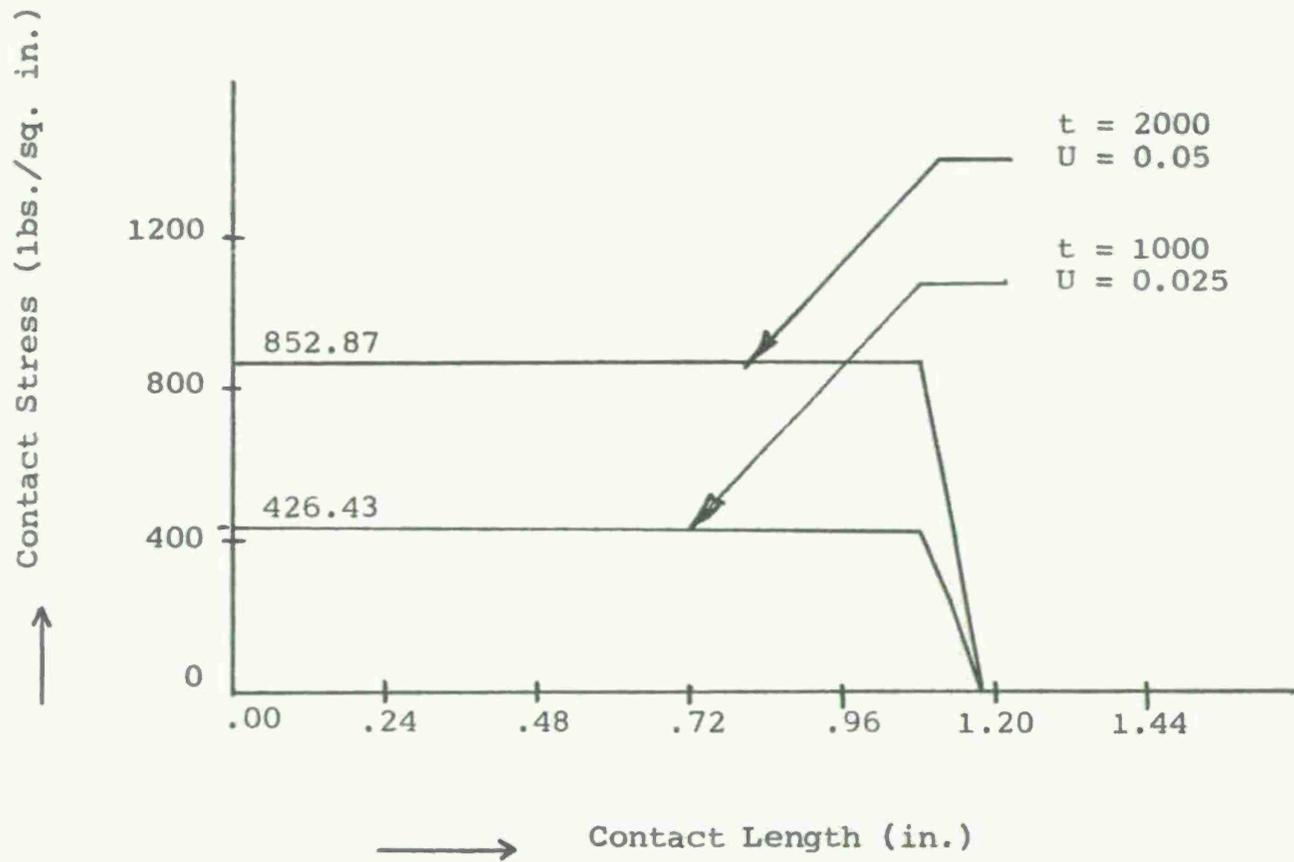


Figure 8. Contact Stress vs. Contact Length with Augmented Cost Function.

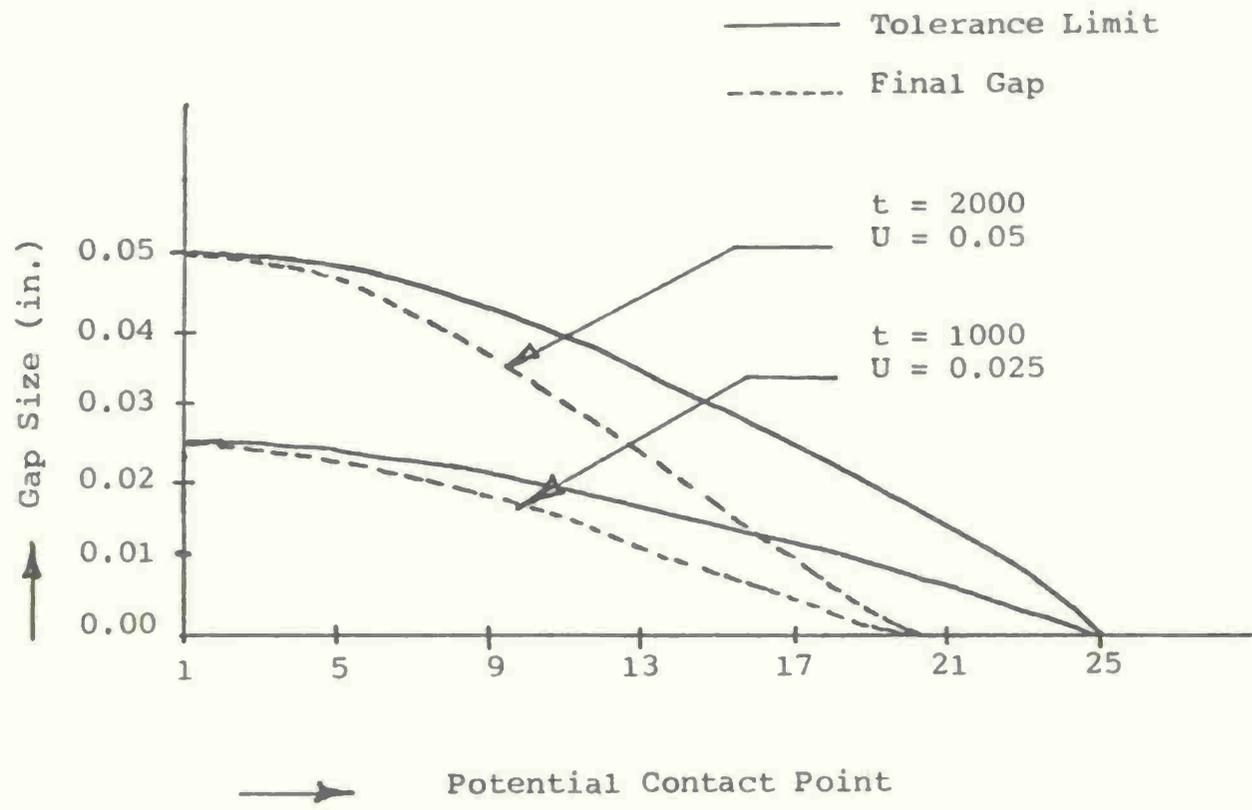


Figure 9. Gap Size vs. Potential Contact Point with Augmented Cost Function.

TABLE 1

COMPARISON OF RESULTS FOR LOADS  
 $t = 1000$  lbs. AND  $t = 2000$  lbs.  
 FOR BEAM ON ELASTIC FOUNDATION

LOAD $t$	SR. NO.	U	PEAK STRESS lbs./sq.in.	COMPUTING TIME (SEC.)	NO. OF ITRNS	TIME/ ITRN (SEC.)	CON- TACT LENGTH (IN.)
1000	1	.01	562.9	38.80	5	7.36	1.86
	2	.015	499.2	48.12	6	8.02	1.98
	3	.02	457.4	72.08	8	9.01	2.22
	4	.025	426.43	91.27	9	10.14	2.34
2000	1	.02	1125.6	36.87	5	7.374	1.86
	2	.03	998.25	48.94	6	8.157	1.98
	3	.04	914.8	73.00	8	9.125	2.22
	4	.05	852.87	93.17	9	10.35	2.34

iterations, with computing time per iteration varying only between 7 and 10 seconds. In all cases, peak stress is reduced considerably and the contact arc increases. For example, for  $t = 1000$  lbs., the peak stress of the non-modified design was  $1120 \text{ lbs./in.}^2$  and contact arc length was 1.38 in. By adjusting the contours of two bodies, with a tolerance limit of  $U = 0.01$  and  $t = 1000$  lbs., Table 2 shows the peak stress reduces to  $552.9 \text{ lbs./sq. in.}$  and the contact arc length becomes 1.86 in.

### 3.2 Initially Bent Beam on an Elastic Foundation

The initial gap between a beam and elastic foundation is given by the formula  $G \left[ 1 - \frac{x^2}{L^2} \right]$ , where  $x$  is the distance from the center of the beam, and  $G$  is a constant greater than zero. Matrices  $B$ ,  $A$ ,  $b$ , and  $C$  are the same as for the problem in the preceding section, except that the vector  $a$  is obtained from the expression given above.

The quadratic programming problem was solved for 25 potential contact points on the half beam, with an interval size of 0.06 in. The potential contact length, as shown in Figure 10(a) is 2.88 in. With  $G = 0.0001$  the resulting contact length was 1.38 in., for loads 1000 lbs. and 2000 lbs. These results are shown in Figure 10(b). The quadratic programming code ZORILLA [10] was again used to solve this problem.

TABLE 2

CONTACT STRESS DISTRIBUTION AND FINAL GAP  
 FOR BEAM ON ELASTIC FOUNDATION  
 $t = 1000 \text{ lbs.}, U = 0.01$

POINT NO.	CONTACT STRESS lb./sq.in.	TOLERANCE LIMIT (in.)	FINAL GAP SIZE FOR MODIFIED CONTOURS (in.)
1	562.886	0.1000000E-01	0.9999964E-02
2	562.888	0.9982634E-02	0.9910598E-02
3	562.889	0.9930551E-02	0.9614620E-02
4	562.891	0.9843744E-02	0.9165816E-02
5	562.891	0.9722218E-02	0.8595929E-02
6	462.892	0.9565968E-02	0.7923093E-02
7	562.892	0.9374995E-02	0.7147454E-02
8	562.894	0.9149302E-02	0.6315298E-02
9	562.893	0.8888885E-02	0.5400788E-02
10	562.893	0.8593746E-02	0.4458770E-02
11	562.895	0.82638863E-02	0.3493937E-02
12	562.888	0.7899299E-02	0.2511400E-02
13	562.891	0.7499997E-02	0.1511019E-02
14	562.885	0.7065967E-02	0.5030558E-02
15	393.626	0.6597217E-02	0.0
16	59.206	0.6093744E-02	0.0
17	0.0	0.5555548E-02	0.0
18	0.0	0.4982639E-02	0.0
19	0.0	0.4374996E-02	0.0
20	0.0	0.3732643E-02	0.0
21	0.0	0.3055555E-02	0.0
22	0.0	0.2343752E-02	0.0
23	0.0	0.1597222E-02	0.0
24	0.0	0.8159771E-03	0.0
25	0.0	0.0	0.0

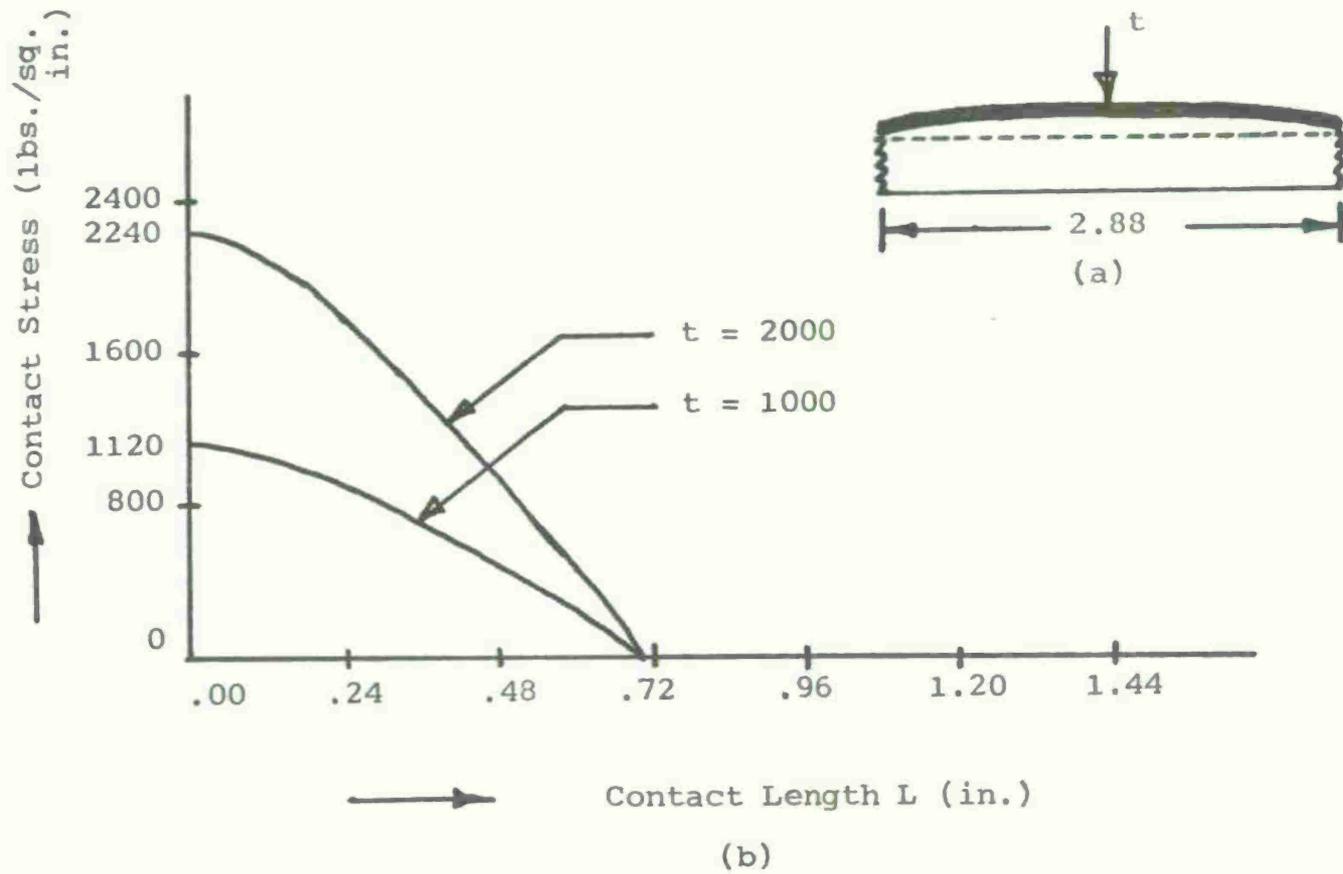


Figure 10. Contact Stress vs. Contact Length from Quadratic Programming Solution, with  $G = 0.0001$ .

One now has the contact arc to be employed for solving the linear programming problem:

$$\text{Minimize } \bar{J} = -b_n + 1$$

subject to constraints

$$\begin{aligned} \hat{S} - b_n + 1 &\leq 0 \\ -\hat{b} &\leq -b^0 \\ \hat{b} &\leq b^1 \\ -\hat{B}\hat{S} - \hat{A}q - \hat{b} &= \hat{a} \\ \hat{A}^T \hat{S} &= c \\ -\tilde{B}\hat{S} - \tilde{A}q &\leq \tilde{a} + \tilde{b} \\ \hat{S} &\geq 0 \end{aligned} \tag{3.2.1}$$

A linear programming tableau is formed from Equations 3.2.1, as explained in the previous section. The linear programming problem was solved and results are presented in Figures (11-16). A comparison of the results for two loads,  $t = 1000$  lbs. and  $t = 2000$  lbs., is given in Table 3. Values of  $U$  used are proportional to applied load for the cases  $t = 1000$  lbs. and  $t = 2000$  lbs. The peak stress obtained for  $t = 1000$  lbs. is exactly half that for  $t = 2000$  lbs. and the contact length is the same for  $t = 1000$  lbs. and  $t = 2000$  lbs. Thus, contact arc length and peak stress are dependent on both value of applied load and  $U$ . Table 4 gives the optimum stress

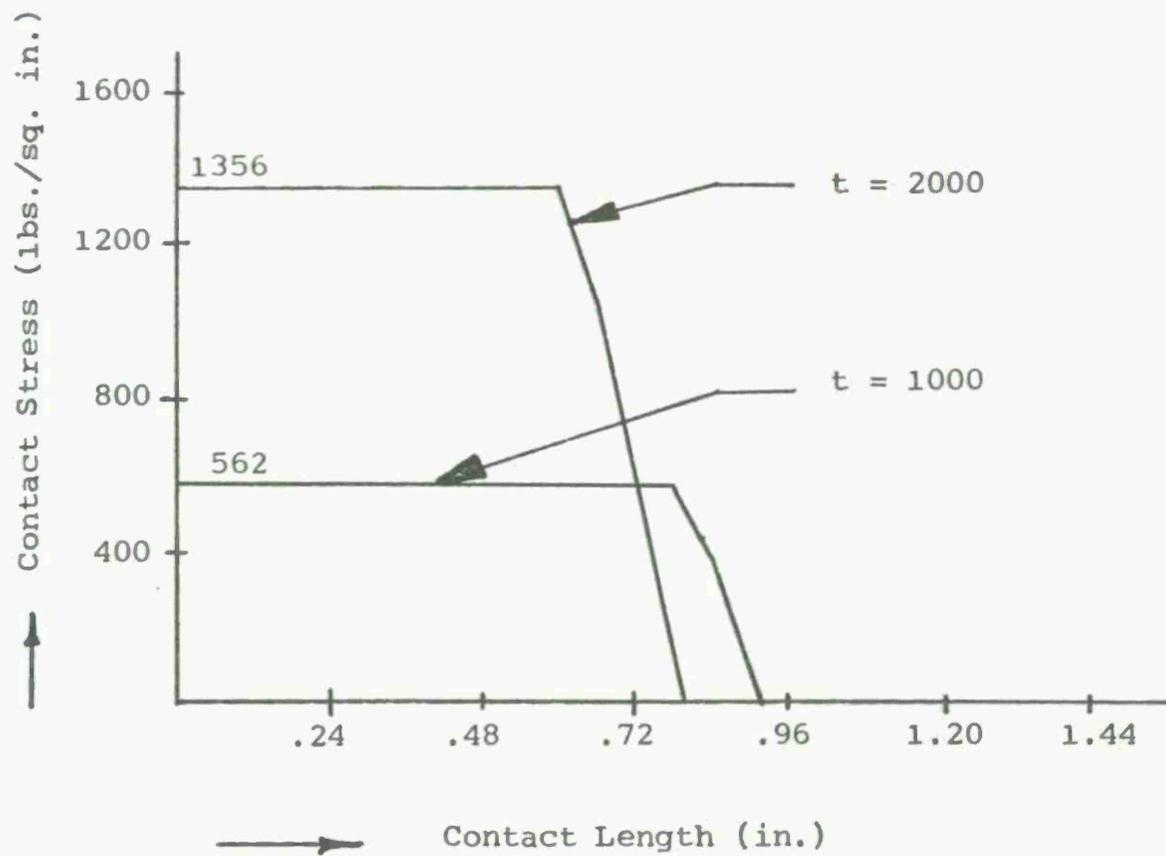


Figure 11. Contact Stress vs. Contact Length with  $U = .01$  and  $G = 0.0001$

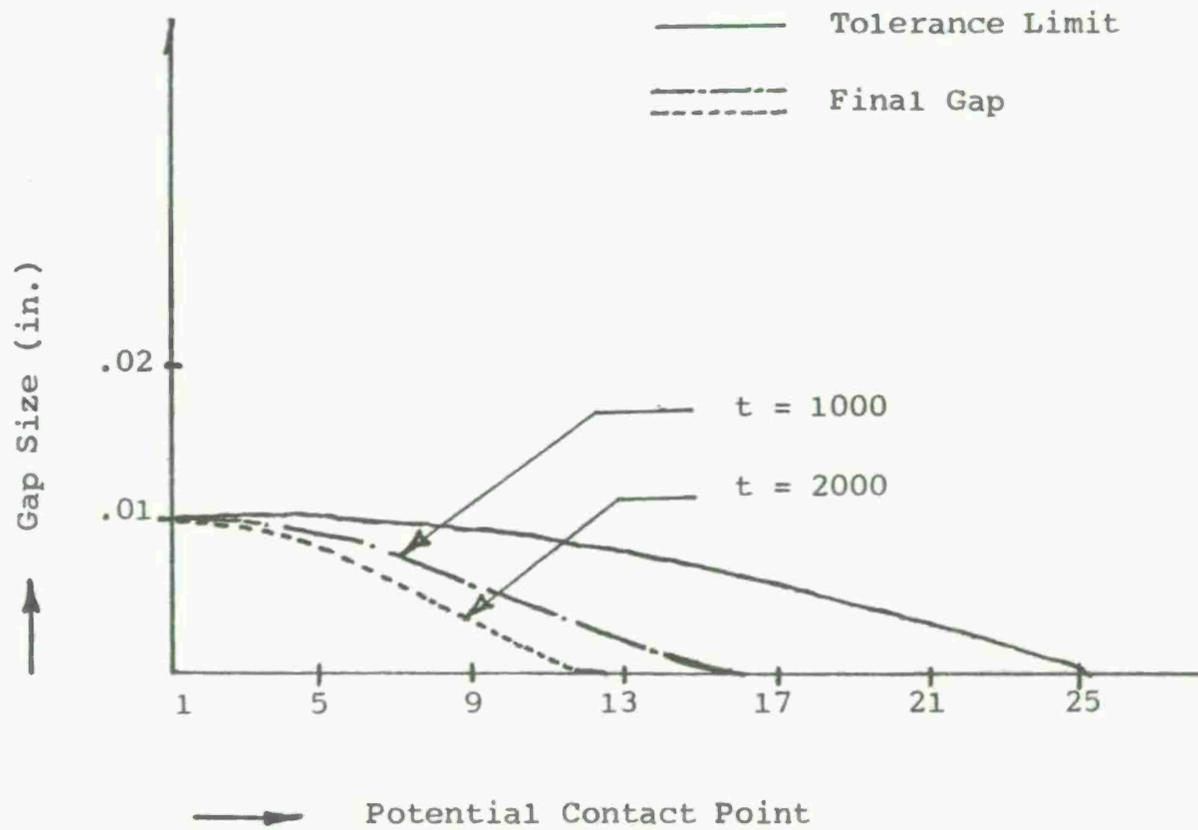


Figure 12. Gap Size vs. Potential Contact Point with  $U = 0.01$  and  $G = 0.0001$ .

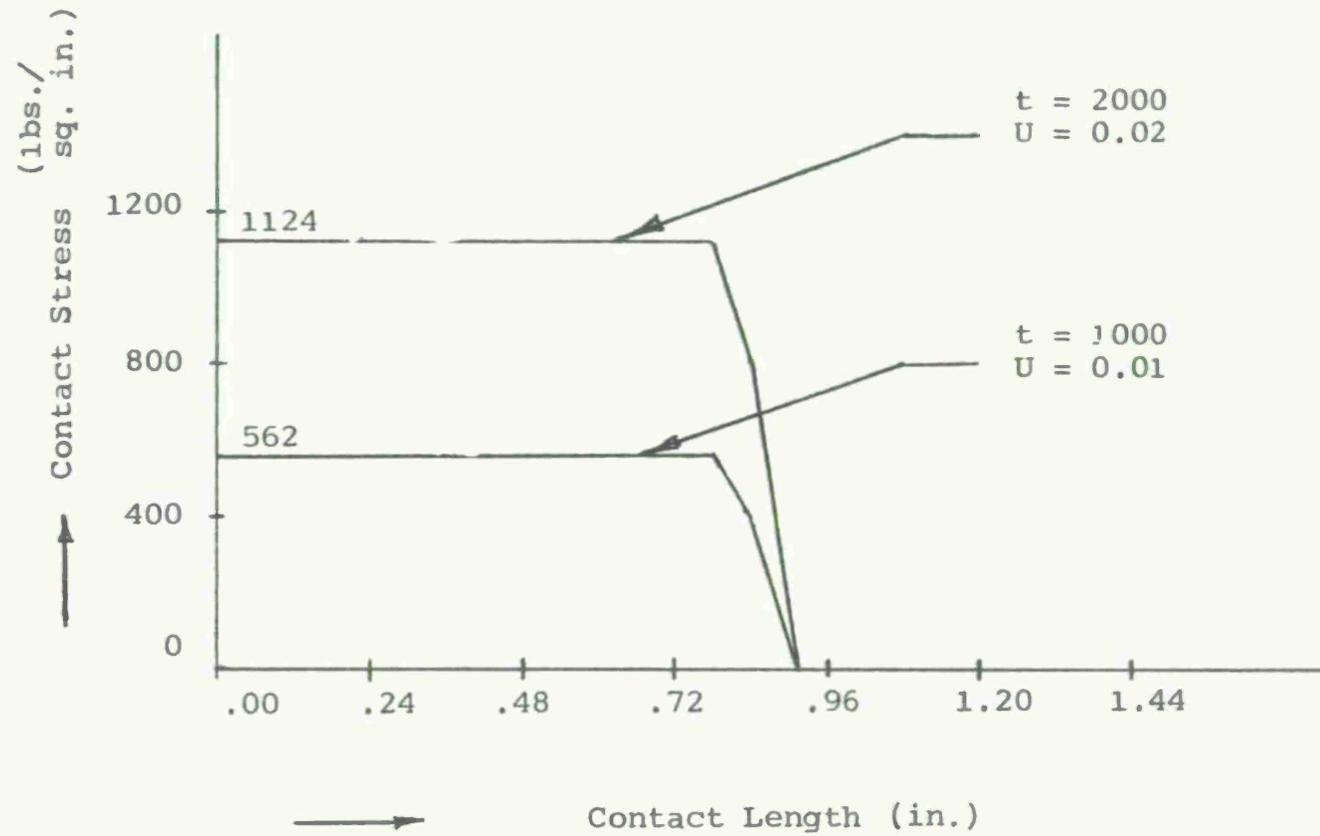


Figure 13. Contact Stress vs. Contact Length with  $G = 0.0001$ .

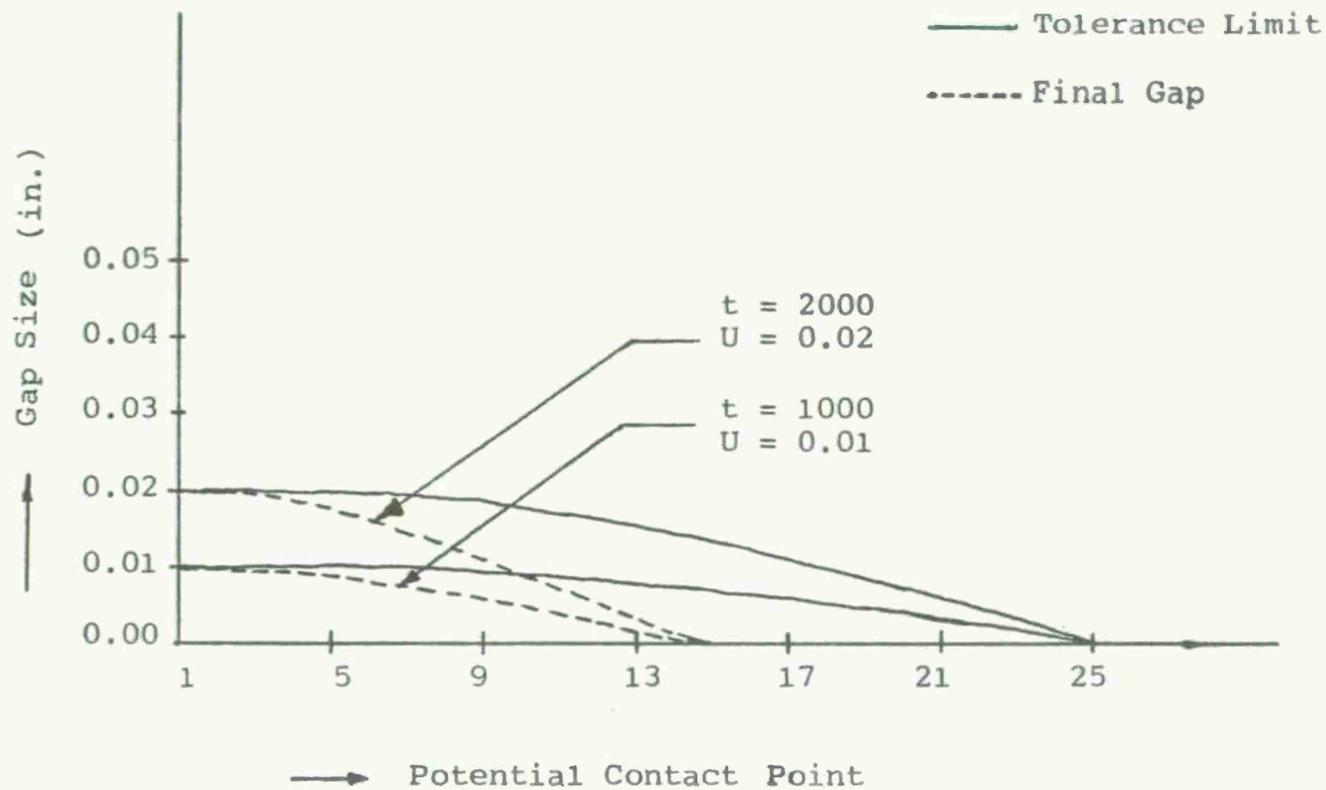


Figure 14. Gap Size vs. Potential Contact Point with  $G = 0.0001$ .

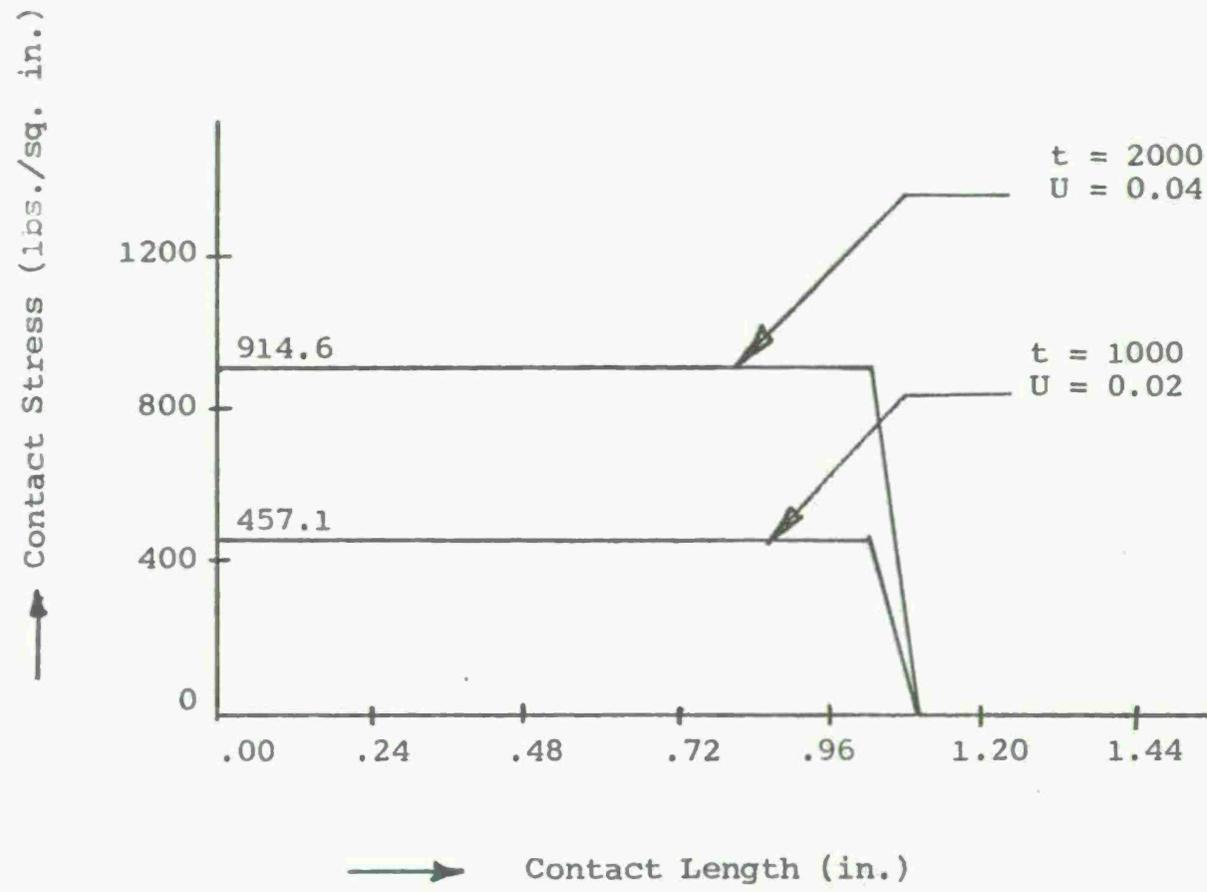


Figure 15. Contact Stress vs. Contact Length with Augmented Cost Function and with  $G = 0.0001$ .

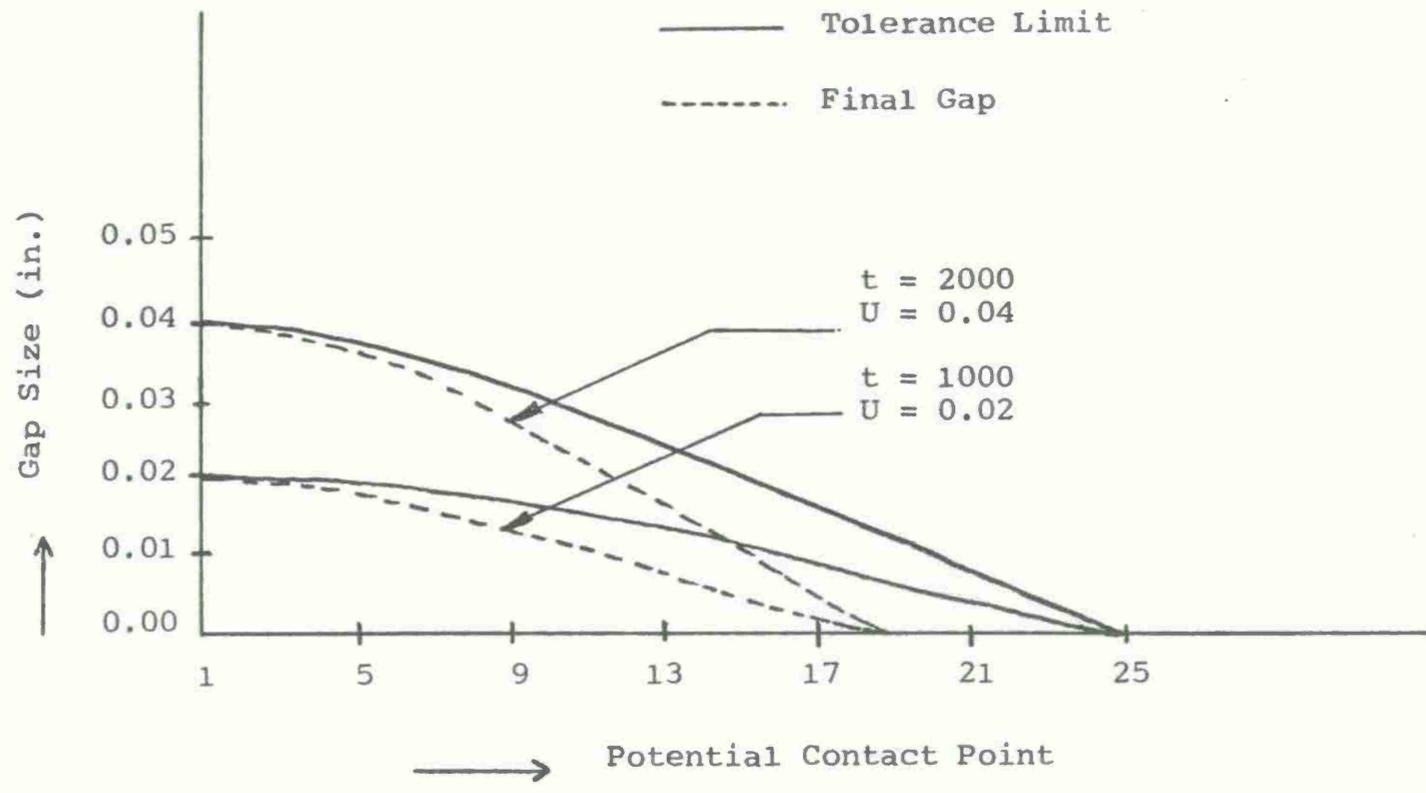


Figure 16. Gap Size vs. Potential Contact Point with Augmented Cost Function and with  $G = 0.0001$ .

TABLE 3

COMPARISON OF RESULTS FOR LOADS  $t = 1000$  lbs.  
AND  $t = 2000$  lbs. FOR AN INITIALLY BENT  
BEAM ON ELASTIC FOUNDATION

LOAD $t$	SR. NO.	U	PEAK STRESS lbs./sq.in.	COMPUTING TIME (SEC.)	NO.OF ITRNS	TIME/ ITRN (SEC.)	CONTACT LENGTH (in.)
1000	1	0.005	677.6	21.90	3	7.3	1.62
	2	0.01	561.7	39.05	5	7.81	1.86
	3	0.015	498.7	49.32	6	8.22	1.98
	4	0.02	457.1	71.82	8	8.98	2.22
2000	1	0.01	1356.0	21.65	3	7.217	1.62
	2	0.02	1124.05	41.37	5	8.274	1.86
	3	0.03	997.2	49.17	6	8.195	1.98
	4	0.04	914.6	73.74	8	9.22	2.22

TABLE 4

CONTACT STRESS DISTRIBUTION AND FINAL GAP FOR  
INITIALLY BENT BEAM ON ELASTIC FOUNDATION  
 $t = 1000$  lbs.,  $U = 0.01$

POINT CONTACT NO.	CONTACT STRESS lb./sq. in.	TOLERANCE LIMIT (in.)	INITIAL GAP (in.)	FINAL GAP (in.)
1	561.712	0.1000000E-01	.1000000E-03	0.9999931E-02
2	561.714	0.9982634E-02	0.9982637E-04	0.9929962E-02
3	561.717	0.9930551E-02	0.9930554E-04	0.9638943E-02
4	561.719	0.9843744E-02	0.9843748E-04	0.9196758E-02
5	561.721	0.9722218E-02	0.9722220E-04	0.8627117E-02
6	561.721	0.9565968E-02	0.9565971E-04	0.7991239E-02
7	561.720	0.9374995E-02	0.9374999E-04	0.7174265E-02
8	561.720	0.9149302E-02	0.9149304E-04	0.6328776E-02
9	561.722	0.8888885E-02	0.8888886E-04	0.5433228E-02
10	561.722	0.8593746E-02	0.8593748E-04	0.4489828E-02
11	561.722	0.8263886E-02	0.8263886E-04	0.3522005E-02
12	561.722	0.7899299E-02	0.7899302E-04	0.2532016E-02
13	561.717	0.7499997E-02	0.7499997E-04	0.1539212E-02
14	561.719	0.7065967E-02	0.7065968E-04	0.5400113E-02
15	406.454	0.6597217E-02	0.6597218E-04	0.0
16	62.750	0.6093744E-02	0.6093745E-04	0.0
17	0.0	0.5555548E-02	0.5555548E-04	0.0
18	0.0	0.4982639E-02	0.4982639E-04	0.0
19	0.0	0.4374996E-02	0.4374996E-04	0.0
20	0.0	0.3732644E-02	0.3732644E-04	0.0
21	0.0	0.3055555E-02	0.3055555E-04	0.0
22	0.0	0.2343752E-02	0.2343752E-04	0.0
23	0.0	0.1597222E-02	0.1597222E-04	0.0
24	0.0	0.8159771E-03	0.8159771E-05	0.0
25	0.0	0.0	0.0	0.0

distribution, tolerance limit, initial gap, and final gap at the potential contact points. Peak contact stress with  $U = 0.01$  and  $t = 1000$  lbs. was found to be 561.712 lbs./sq. in. This peak stress is not much different from the one obtained in the case of beam on elastic foundation with no initial gap.

The quadratic programming problem was again solved for the same data, but with an initially bent beam on an elastic foundation with  $G = 0.001$  and  $G = 0.005$ . The results are presented in Figures (17-18). It is observed that, as the value of  $G$  is increased, peak contact stress decreases. In the case of  $G = 0.005$  contact length for 1000 lbs. and 2000 lbs. loads differs. It is also observed that peak contact stress depends on the applied load.

Once the contact arc is known from the contact analysis problem, the linear programming problem was solved for  $G = 0.001$  and  $G = 0.005$ , with two different loads in each case. Results are presented in Figures (19-22). Table 5 shows the comparison of results with  $U = 0.01$ , for a beam on an elastic foundation with no initial gap and with an initial gap. Table 6 shows the comparison of results for  $t = 2000$  lbs. with values of  $U$  ranging from 0.01 to 0.05. It is noted that the peak contact stress decreases if the value of  $U$  and  $G$  are increased.

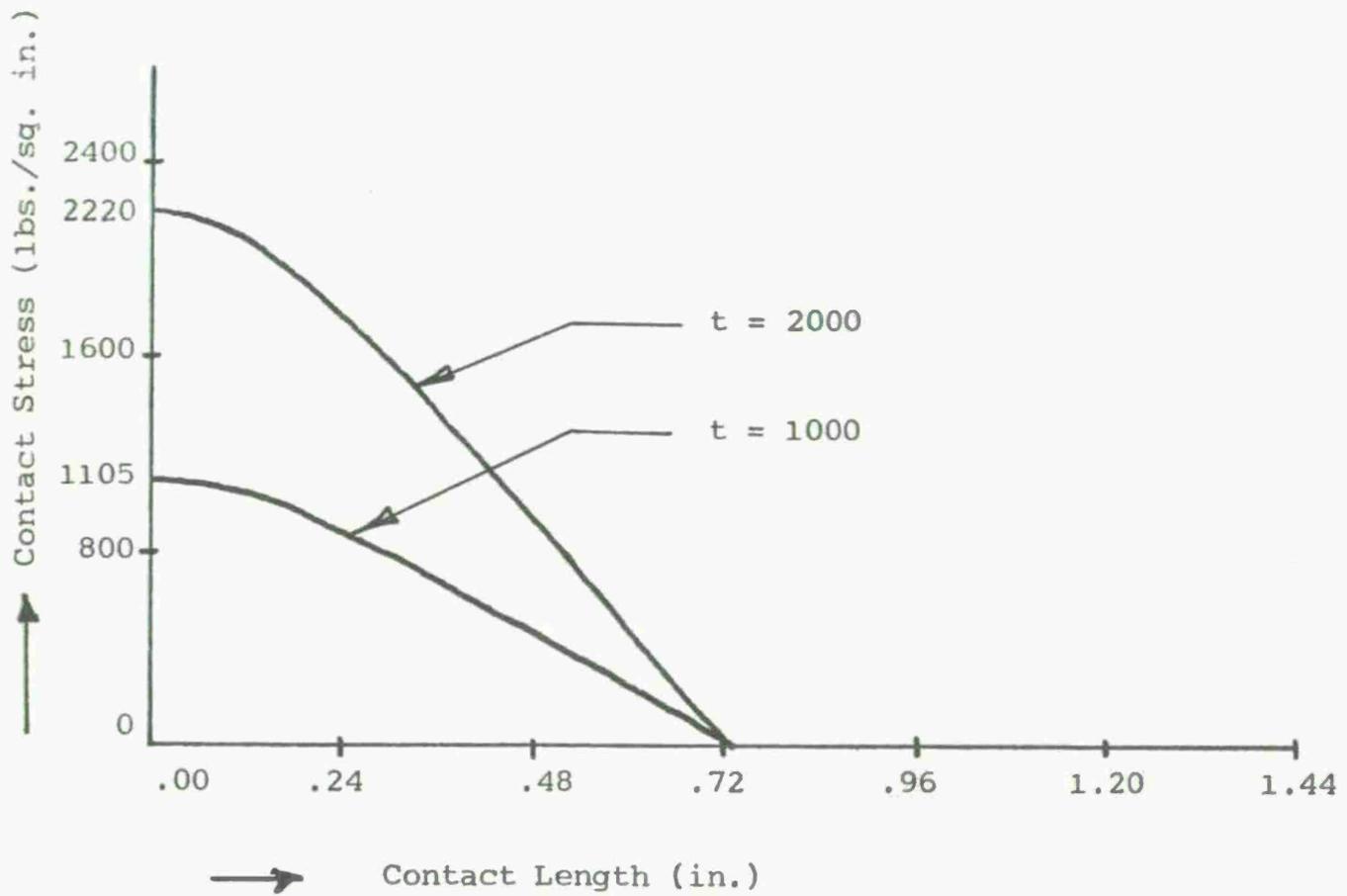


Figure 17. Contact Stress vs. Contact-Length from Quadratic Programming Solution with  $G = 0.001$ .

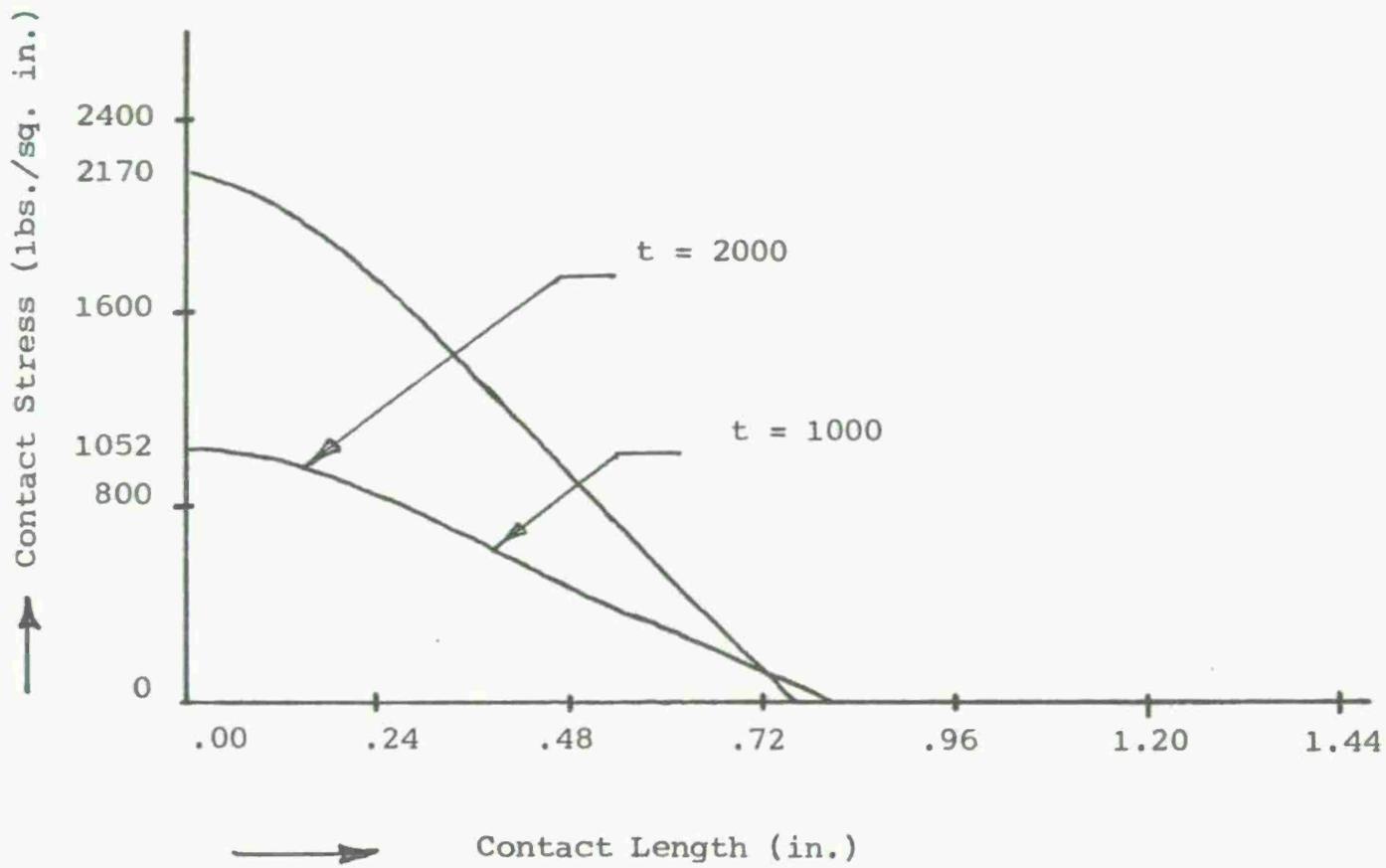


Figure 18. Contact Stress vs. Contact Length from Quadratic Programming Solution with  $G = 0.005$ .

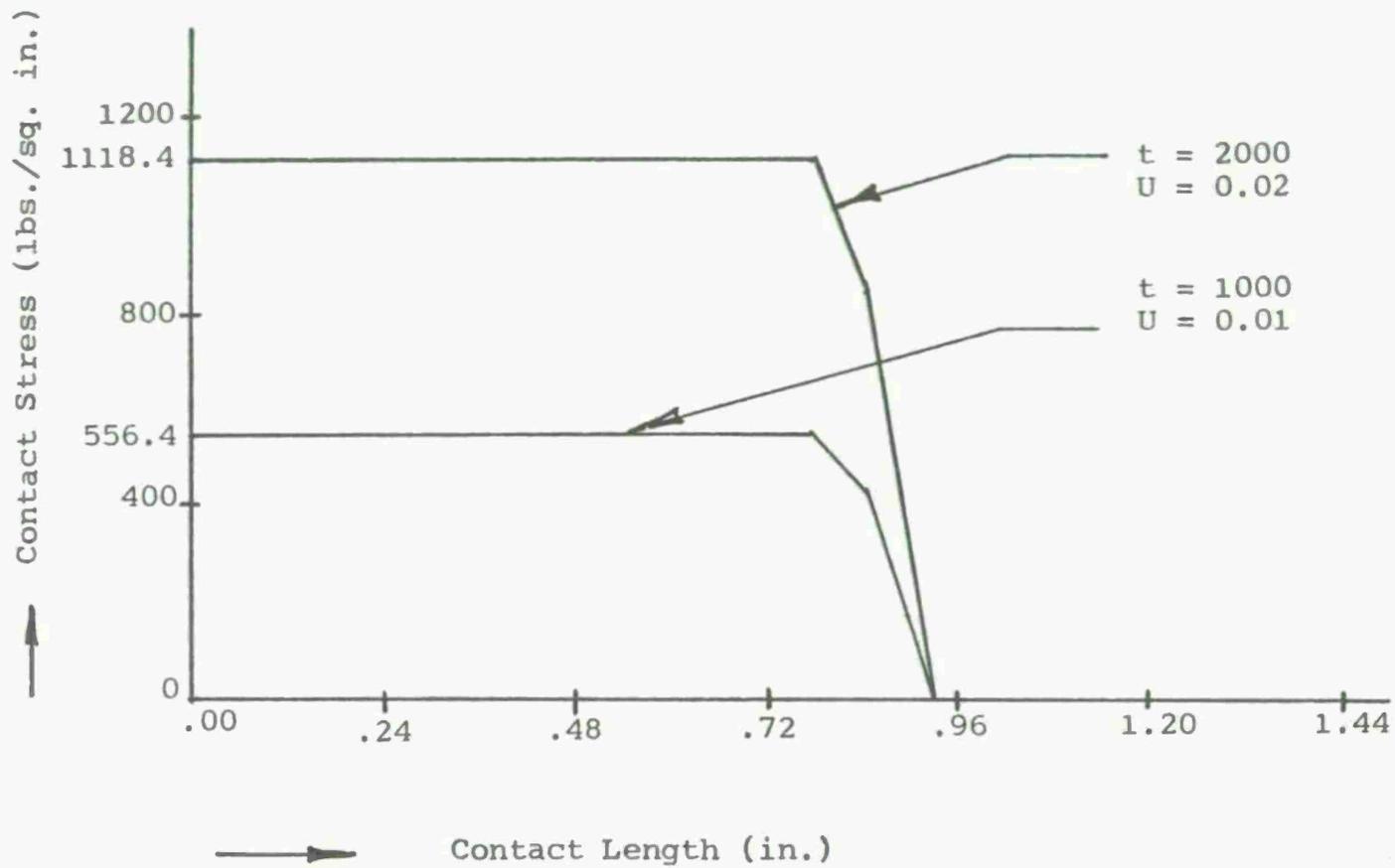


Figure 19. Contact Stress vs. Contact Length with  $G = 0.001$ .

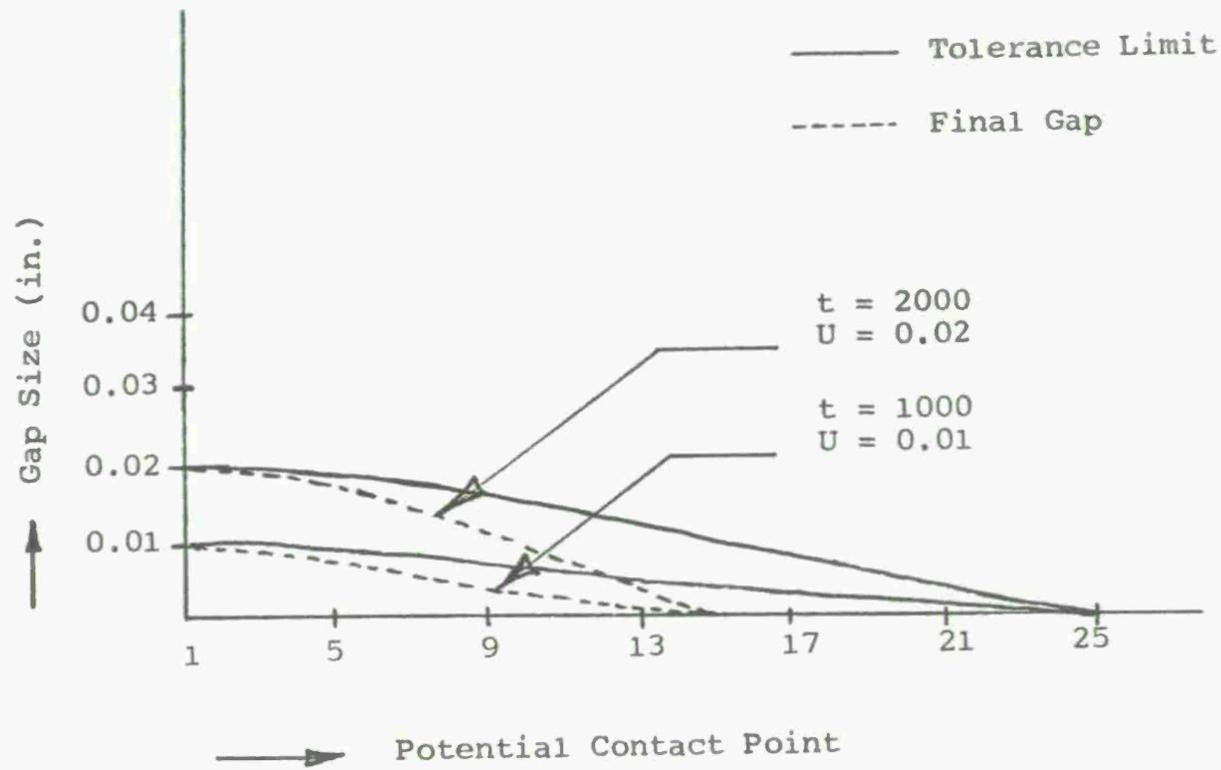


Figure 20. Gap Size vs. Potential Contact Point with  $G = 0.001$ .

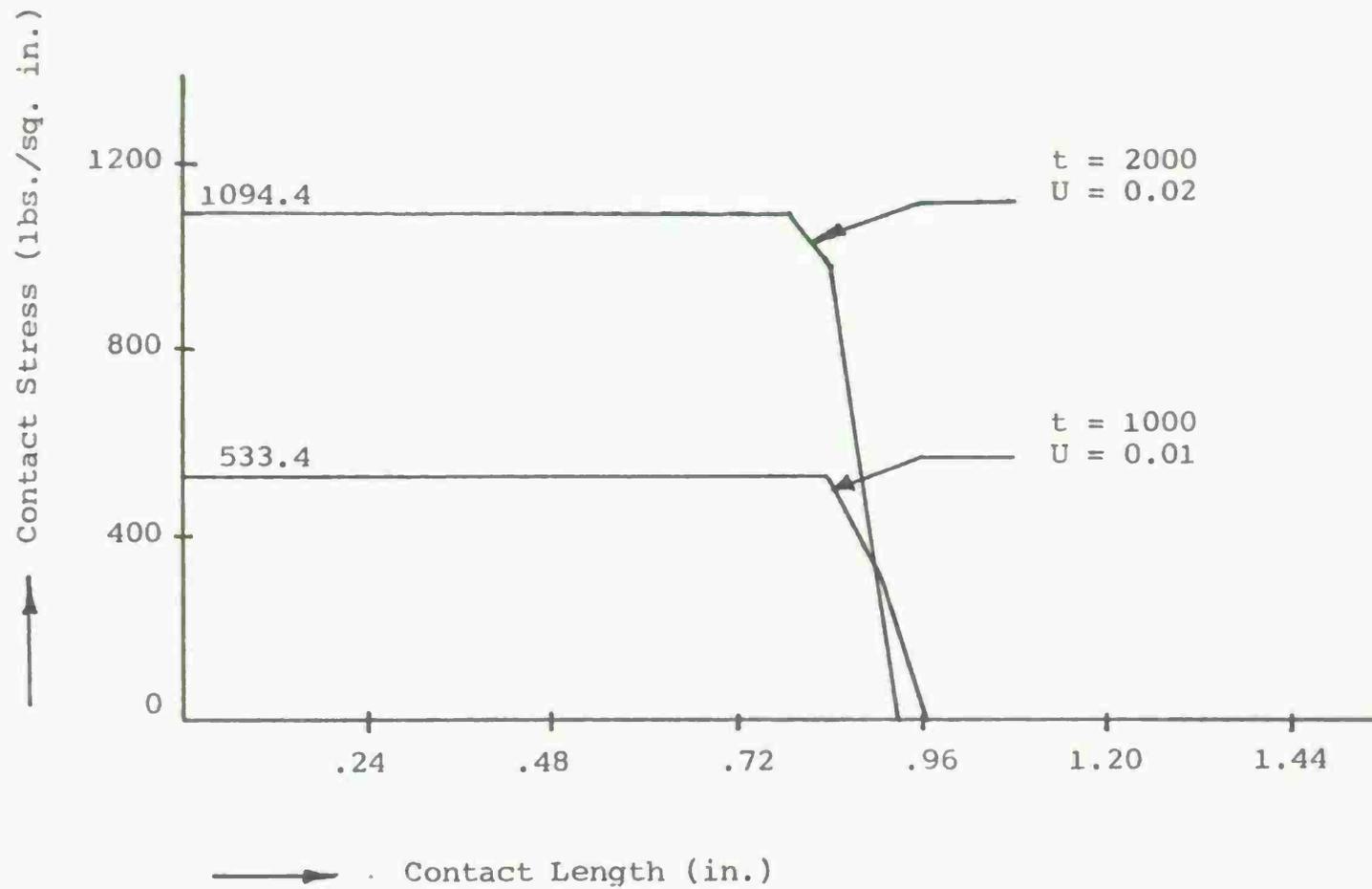


Figure 21. Contact Stress vs. Contact Length with  $G = 0.005$ .

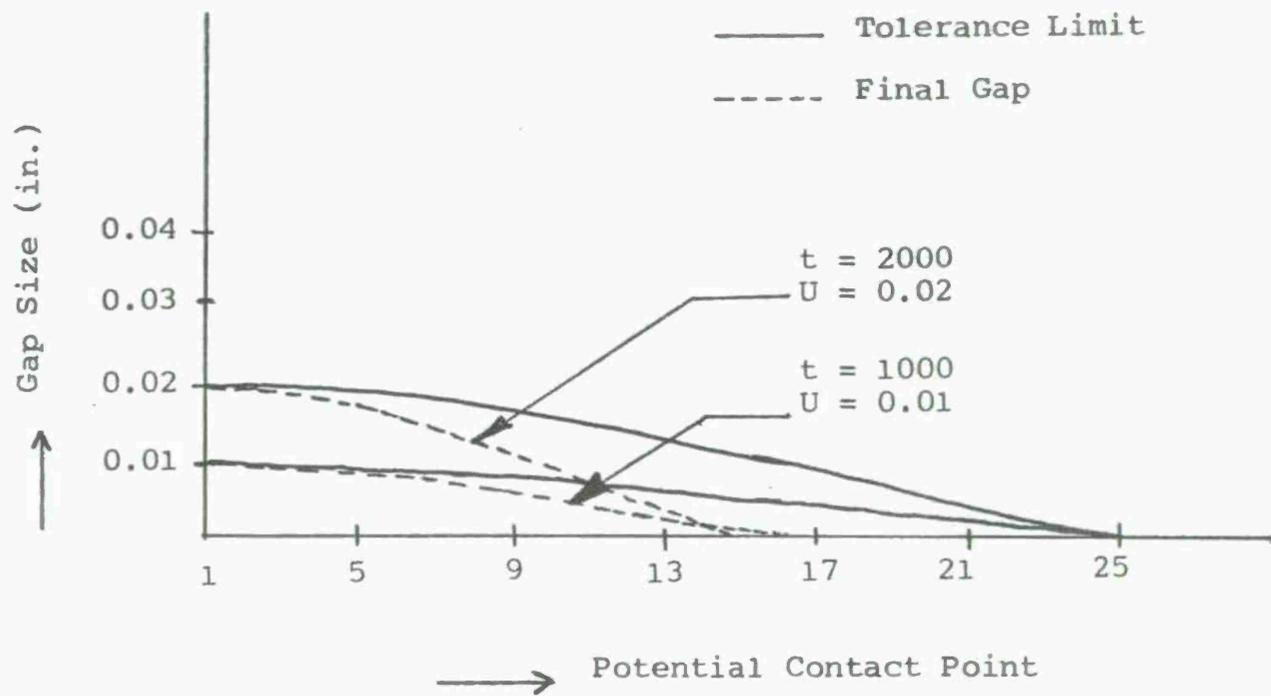


Figure 22. Gap Size vs. Potential Contact Point with  $G = 0.005$ .

TABLE 5

COMPARISON OF RESULTS FOR BEAM ON ELASTIC FOUNDATION  
WITH NO INITIAL GAP AND WITH DIFFERENT  
INITIAL GAPS FOR  $U = 0.01$

LOAD (lbs.)	G	CONTACT STRESS (lbs./ sq. in.)	CONTACT LENGTH (in.)	NUMBER OF ITRATIONS	TIME/ITRN (sec.)
1000	0	562.9	1.86	5	7.36
	.0001	561.7	1.86	5	7.81
	.001	556.4	1.86	4	8.6
	.005	533.4	1.98	4	9.375
2000	0	1358.5	1.62	3	7.25
	.0001	1356.0	1.62	3	7.22
	.001	1349.5	1.62	2	7.92
	.005	1317.8	1.62	2	8.5

TABLE 6

COMPARISON OF RESULTS FOR BEAM ON ELASTIC FOUNDATION  
WITH NO INITIAL GAP AND WITH DIFFERENT INITIAL  
GAPS FOR  $t = 2000$  lbs.

G	ITEM	QP SOLUTION	LINEAR PROGRAMMING SOLUTION				
			U				
			0.01	0.02	0.03	0.04	0.05
0		2242	1358.5	1125.6	993.25	914.8	852.87
.0001	CONTACT	2240	1356	1124	997.2	914.6	852.6
.001	STRESS	2220	1349.5	1118.4	994.5	911.1	849.2
.005	lbs./in. <sup>2</sup>	2170	1317.8	1094	973.3	895.2	833.9
0		1.38	1.62	1.86	1.98	2.22	2.34
.0001	CONTACT	1.38	1.62	1.86	1.98	2.22	2.34
.001	LENGTH	1.50	1.62	1.86	1.98	2.22	2.34
.005	in.	1.50	1.62	1.86	2.10	2.22	2.34

## Chapter IV

## CONCLUSIONS

The technique presented here to solve surface contour design problem is quite simple, and has given consistently good results. By adjusting the contours of the contacting bodies, the stresses developed are reduced considerably. The problem of a beam on an elastic foundation demonstrates some of these facts. The peak stress of the unmodified structure is highest at the center of the beam and decreases at the end points of the contact region. The contact design problem gives a relatively constant stress distribution on the contact region, with much reduced peak contact stresses.

Tolerance limits affect the solution as is shown in Table 6. In this problem, the load and design tolerance limits have a measurable effect on the solution of the design problem. However, as shown in Table 1, the solution changes proportionally if the load and tolerance limit are changed in the same proportion.

Suggestions

This design method can be extended to other problems as follows:

1. Asymmetrical problems for beam on an elastic foundation.
2. Circular inclusion problems for elastic bodies in contact.
3. Multibody contact problems.

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APPENDICES

## APPENDIX A:

FORMULATION OF MATRICES B, A, a, and C

The compatibility condition for deformation is given by

$$\mathcal{E} = u^1 + u^2 + d \geq 0 \quad (\text{A-1})$$

where  $\mathcal{E}$  is the gap vector after deformation of two bodies,  $u^1$  and  $u^2$  denote the normal displacement vectors of potential contact points on Bodies 1 and 2, and  $d$  is the vector of initial gap between the two bodies.

Body 2 of Figure 23 is fixed, so only rigid body displacement for Body 1 is considered. The displacement of points on Body 1 is due to rigid body displacements and elastic deformation. Elastic deformation of points on Body 1 is determined relative to zero values of the rigid body coordinates, and total displacement is found by superposition. Generalized displacement coordinates are denoted by  $q$ .

The total displacement of potential contact points on Body 1 can be written in vector form as

$$u^1 = P^T u_e^1 + Aq \quad (\text{A-2})$$

where  $u_e^1$  is a reduced vector of elastic displacements of potential contact points on Body 1. The vector  $u_e^1$

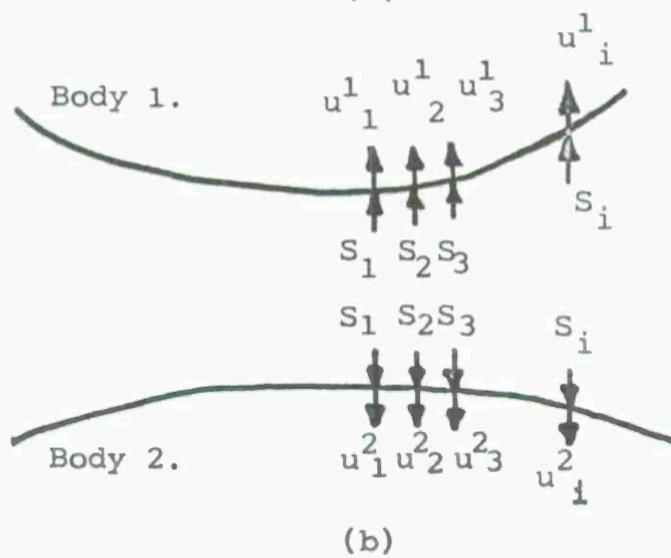
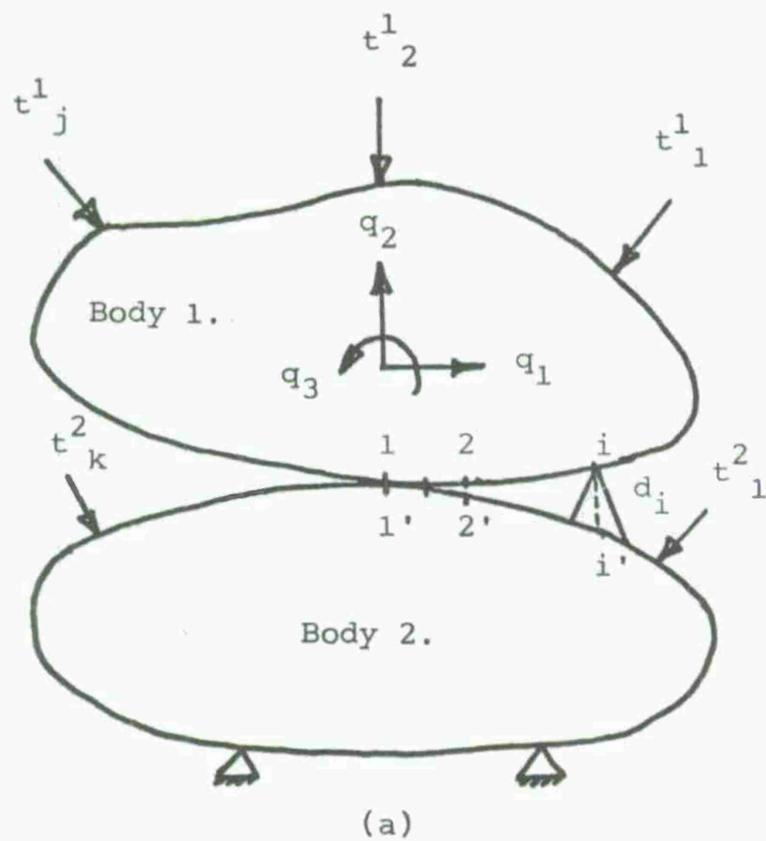


Figure 23. Two Bodies in Contact.

is determined with  $q = 0$ , so it cannot contain displacement components that are used as rigid body degrees of freedom. The projection matrix  $P$  is given by  $P = [0 \mid I_{n-m}]$ , where  $n$  is the number of potential contact points and  $m$  is the number of degrees of freedom. The matrix  $A$  is an  $n \times m$  affine transformation that contributes to rigid body displacements.

The deformation vector  $u_e^1$  is further decomposed as

$$u_e^1 = F^1 P S + V_e^1 \quad (\text{A-3})$$

where  $F^1$  is the flexibility matrix for Body 1, with  $q = 0$ , and  $V_e^1$  is the vector of elastic displacements of potential contact points due to externally applied forces  $t^1$  on Body 1, with  $q = 0$ . From equations (A-2) and (A-3),

$$u^1 = P^T [F^1 P S + V_e^1] + A q \quad (\text{A-4})$$

For Body 2,

$$u^2 = F^2 S + V_e^2, \quad (\text{A-5})$$

where  $F^2$  is an  $n \times n$  flexibility matrix for potential contact points on Body 2, and  $V_e^2$  is displacement of potential contact points on Body 2, due to externally

applied forces.

Equation (A-1) can now be written as

$$\mathcal{E} = (P^T F^1 P + F^2) S + Aq + P^T V^1 e + V^2 e + d \geq 0, \quad (A-6)$$

so

$$B = P^T F^1 P + F^2$$

$$A \equiv A$$

and

$$a = P^T V^1 e + V^2 e + d$$

#### Equilibrium Equations

The work done by stresses and the applied forces on Body 1, due to rigid body displacements only, is

$$W = S^T Aq + t^1{}^T Hq \quad (A-7)$$

where  $H$  is a matrix that gives rigid body displacement at the points of application of  $t^1$ .

Varying the rigid body displacements, the principle of virtual work gives

$$\delta W = S^T A \delta q + t^1{}^T H \delta q = 0 \quad (A-8)$$

Thus,

$$(S^T A + t^{1T} H) \delta q = 0 \quad (A-9)$$

Since all components of  $q$  are independent, it is necessary that

$$A^T S + H^T t^1 = 0 \quad (A-10)$$

or

$$A^T S = -H^T t^1$$

Defining

$$C = -H^T t^1 \quad (A-11)$$

Equation (A-10) can be written as

$$A^T S = C \quad (A-12)$$

APPENDIX B:

PROGRAM LISTING

```

C *****
C *
C *                               *
C *           MAIN PROGRAM           *
C * BEAM ON AN ELASTIC FOUNDATION WITH NO INITIAL GAP. *
C *                               *
C *****
C
C
C           DESCRIPTION OF PARAMETERS.
C
C DEL      IS THE INTERVAL SIZE BETWEEN THE POTENTIAL CONTACT
C           POINTS(P.C.P.)
C AL       IS HALF THE P.C.LENGTH.
C MNM      IS THE NO. OF DEGREES OF FREEDOM.
C NN       IS THE NO. OF P.C.PS.
C NC=NCI   IS THE NO. OF POINTS IN CONTACT.
C NNC      IS THE NO. OF POINTS NOT IN CONTACT.
C P        IS THE EXTERNALLY APPLIED LOAD.
C ITRN     IS THE LINEAR PROGRAMMING ITERATION NUMBER.
C M1       IS THE NO. OF CONSTRAINTS PLUS ONE.
C N1       NO. OF DESIGN VARIABLES PLUS NO.OF GE. TYPE OF CONSTRAINTS.
C
C M1=1+4*NC+MNM+NNC
C N1=2*MNM+1+2*NC+NGE
C
C           DESCRIPTION OF MATRICES.
C
C A1(NN,MNM) IS THE AFFINE TRANSFORMATION MATRIX FOR RIGID BODY
C            DISPLACEMENT CONTRIBUTION FOR BODY 1.
C T(NN,NN)   IS THE COMBINED INFLUENCE COEFFICIENT MATRIX.
C F(NN,NN)   IS EQUIVALENT TO T(NN,NN).
C
C FOLLOWING MATRICES ARE GENRATED FROM THE ABOVE TWO MATRICES.
C

```

C BHAT(NN,NN) IS THE REDUCED MATRIX 'T' WITH ROWS AND COLUMNS  
 C CORRESPONDING TO POINTS NOT IN CONTACT REMOVED.  
 C AHAT(NN,MNM) IS THE MATRIX 'A1' WITH CORRESPONDING ROWS FOR  
 C POINTS NOT IN CONTACT REMOVED.  
 C BTEL(NN,NN) IS THE MATRIX 'T' WITH ROWS CORRESPONDING TO POINTS  
 C IN CONTACT AND COLUMNS CORRESPONDING TO POINTS NOT  
 C IN CONTACT REMOVED.  
 C ATEL(NN,MNM) IS THE MATRIX 'A1' WITH ROWS CORRESPONDING TO  
 C POINTS IN CONTACT REMOVED.  
 C B(M1,N1) IS THE MATRIX OF CONSTRAINT COEFFICIENTS.

DESCRIPTION OF VECTORS.

C SA(NN) IS THE VECTOR OF TOLRANCE LIMIT FOR CONTOUR  
 C DESIGN VARIABLES.  
 C SSA(NN) IS THE VECTOR OF INITIAL GAP BETWEEN THE TWO BODIES  
 C AT THEIR P.C.PS.  
 C SB(NN) IS THE VECTOR OF CONTOUR MODIFICATIONS.  
 C SAHAT(NN) IS THE VECTOR 'SA' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT REMOVED.  
 C ASHAT(NN) IS THE VECTOR 'SSA' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT REMOVED.  
 C ASHAT(NN) IS THE VECTOR 'SB' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT REMOVED.  
 C SATEL(NN) IS THE VECTOR 'SA' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT.  
 C SBTEL(NN) IS THE VECTOR 'SA' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT.  
 C ASTEL(NN) IS THE VECTOR 'SSA' WITH COMPONENTS CORRESPONDING TO  
 C POINTS NOT IN CONTACT.  
 C RQ(M1) IS THE VECTOR OF R.H.S. VALUES OF CONSTRAINTS.  
 C NBP(N1) COST FUNCTION ROW VECTOR.  
 C K11(NN) IS THE VECTOR OF INDICED FOR POINTS NOT IN CONTACT.  
 C K(NN) IS THE VECTOR OF INDICES OF POINTS IN CONTACT.

C KII(NN) IS EQUIVALENT TO K(NN).  
 C II(NN) IS THE VECTOR HAVING COMPONENTS AS 1 OR 0.  
 C 1 FOR POINTS IN CONTACT AND 0 FOR PTS. NOT IN CONTACT.  
 C SHAT(NN) IS THE VECTOR OF CONTACT FORCES.  
 C RBD(MNM) IS THE RIGID BODY DISPLACEMENT VECTOR.  
 C RBD1(2\*MNM)  
 C Z(MNM) IS THE VECTOR HAVING COMPONENTS AS TOTAL EXTERNAL  
 C APPLIED LOAD.  
 C X(I) IS THE DISTANCE OF I TH P.C.P. FROM CENTER.  
 C CST(NN) IS THE CONTACT STRESS VECTOR.  
 C SIGN(M1) INDICATES THE TYPE OF CONSTRAINT USED.  
 C -1 FOR GE. TYPE OF CONSTRAINT.  
 C 0 FOR EQ. TYPE OF CONSTRAINT.  
 C 1 FOR LE. TYPE OF CONSTRAINT.

C NOTE THAT

C THE DIMENSIONS OF THE MATRICES AND VECTORS SHOULD BE GIVEN AS  
 C SHOWN IN THE BRACKETS. ACTUAL SIZE OF 'HAT' AND 'TELDA' MATRICES  
 C AND VECTORS IS GENERALLY LESS.  
 C HAT AND TEL ATTACHED TO THE MATRICES AND VECTORS  
 C DENOTES  $\wedge$  AND  $\sim$  RESPECTIVELY.

C INPUT DATA FOR MAIN PROGRAM OF LP SUBROUTINE.

C 1ST SET OF CARDS. READ IN M1,N1  
 C 2ND SET OF CARDS. READ IN OR GENERATE MATRIX 'B'  
 C 3RD SET OF CARDS. READ IN OR GENERATE VECTOR 'RQ'.  
 C 4TH SET OF CARDS. READ IN SIGN ACCORDING TO TYPE OF CONSTRAINT.

C IMPORTANT NOTE.

```

C
C
C   IN CASE OF BEAM ON ELASTIC FOUNDATION WITH INITIAL GAP
C   A FEW CHANGES IN THE MAIN PROGRAM ARE NECESSARY.
C
C   VECTOR SSA IS CHANGED.
C
C   SSA IN THE EXAMPLE SOLVED IN THIS WORK IS TAKEN AS GIVEN BELOW.
C
C   SSA(I)=0.0001*(1.-((X(I))**2)/(AL**2))
C
C   FEW CHANGES IN MATRIX 'B' ARE AS FOLLOWS.
C
C   B(I+3*NC,K1)=-AHAT(I,K1)
C 320 B(I+3*NC,K1+MNM)=AHAT(I,K1)
C     B(I+3*NC,MM+I)=-1.
C 330 B(I+3*NC,MM+NC+K1)=-BHAT(I,K1)
C
C
C   INTEGER SIGN,GP,PG
C   REAL NBP
C
C   COMMON B(105,105),RQ(105),NBP(105),SIGN(105),KK(105),ITRN
C   COMMON RRQ(105),KN,GP,PG
C
C   DIMENSION SHAT(25),SA(25),ASHAT(25),SB(25),K(25),II(25)
C   DIMENSION KI1(25),BHAT(25,25),SAHAT(25),AHAT(25,2),SSA(25)
C   DIMENSION BTEL(25,25),ATEL(25,2),SATEL(25),SBTEL(25),ASTE(25)
C   DIMENSION K11(25),P(105),RBD(2),RBD1(4)
C   DIMENSION T(25,25),A1(25,2),Z(2),SBHAT(25),F(25,25),X(25),CST(25)
C
C   READ(5,3) PP,U,AL,DEL,MNM,NN,NC,NNC,NGE
C
C   3 FORMAT(4F10.4,5I5)

```

```

      Z(1)=PP
      Z(2)=0.0
      DO 5 J=1,MNM
C
      5 READ(5,2)(A1(I,J),I=1,NN)
C
      DO 20 I=1,NN
20  SSA(I)=0.
      DO 1 I=1,NN
      X(I)=(I-1)*DEL
      1 SA(I)=U*(1.-(((X(I))**2)/(AL**2)))
      DO 4 I=1,NN
      SB(I)=0.
C
      4 READ(5,2)(F(I,J),J=1,NN)
C
      2 FORMAT(5E15.7)
      DO 22 I=1,NN
      DO 22 J=1,NN
22  T(I,J)=F(I,J)
      ITRN=1
C
      K(NC) EQUIVALENT TO KI1(NC) IS THE INDEX OF POINTS IN CONTACT.
C
C
C
C
      READ(5,40)(K(I),I=1,NC)
C
      40 FORMAT(40I2)
      DO 25 I=1,NC
      25 KI1(I)=K(I)
      725 CCNTINUE
      JJ=0
      DO 35 I=1,NC
      35 K(I)=KI1(I)

```

```
K0=0
NM=0
K2=0
N12=0
DC 42 I=1,NN
42 II(I)=0
DC 60 I=1,NN
DC 70 NN1=1,NC
L=K(NN1)
IF(I-L)69,100,69
100 KC=K0+1
II(L)=1
```

C  
C  
C  
C

TO FORM MATRIX B-'HAT', A-'HAT', SMALL-A-'HAT', SMALL-B-'HAT',  
AND VECTOR OF INITIAL GAP FOR POINTS IN CONTACT.

```
DO 220 KUI=1,MNM
220 AHAT(KO,KUI)=A1(L,KUI)
SAHAT(KO)=SA(L)
SBHAT(KO)=SB(L)
ASHAT(KO)=SSA(L)
DC 71 J=1,NN
DO 71 KM=1,NC
M3=K(KM)
IF(J-M3)71,200,71
200 NM=NM+1
BHAT(KO,NM)=T(I,J)
71 CCNTINUE
NM=0
GO TO 70
69 K2=K2+1
IF(NC-K2)70,80,70
80 JJ=JJ+1
```

C

C TO FORM MATRIX B-TELDA(BTEL),A-TELDA(ATEL),SMALL-A-TELDA  
C (ATEL),SMALL-B-TELDA(BTEL),AND VECTOR OF INITIAL GAP FOR  
C POINTS NOT IN CONTACT.  
C

DO 222 MZ=1,MNM  
222 ATEL(JJ,MZ)=A1(I,MZ)  
SATEL(JJ)=SA(I)  
SBTEL(JJ)=SB(I)  
ASTELE(JJ)=SSA(I)  
K11(JJ)=I  
DC 72 J=1,NN  
DC 72 KM3=1,NC  
M2=K(KM3)  
IF(J-M2)72,250,72  
250 N12=N12+1  
BTEL(JJ,N12)=T(I,J)  
72 CCNTINUE  
N12=0  
70 CCNTINUE  
K2=0  
60 CONTINUE  
101 FCRMAT(/5E15.7/)  
N1=2\*MNM+1+2\*NC+NCE  
N=2\*MNM+1+2\*NC  
M1=1+4\*NC+MNM+NNC

C FORMULATION OF 'B' MATRIX.  
C FCRMULATION OF VECTOR 'RQ'.  
C

DO 300 I=1,M1  
RQ(I)=0.  
DC 300 J=1,N1  
300 B(I,J)=0.

C

C COST FUNCTION ROW FOR MAXIMIZATION PROBLEM(NBP).

C

```
DO 310 I=1,N1
310 NBP(NGE+I)=0.
MM=2*MNM+1
NBP(NGE+MM)=-1.0
DO 350 I=1,NC
B(I,MM)=-1.0
B(I,MM+NC+I)=1.0
B(I+NC,MM+I)=1.
B(I+2*NC,MM+I)=-1.
DO 320 K1=1,MNM
B(I+3*NC,K1)=AHAT(I,K1)
320 B(I+3*NC,K1+MNM)=-AHAT(I,K1)
B(I+3*NC,MM+I)=1.
DO 330 K1=1,NC
330 B(I+3*NC,MM+NC+K1)=BHAT(I,K1)
DO 340 K1=1,MNM
340 B(4*NC+K1,MM+NC+I)=AHAT(I,K1)
350 CCNTINUE
DO 360 KJ=1,NNC
DO 355 K1=1,MNM
B(KJ+4*NC+MNM,K1)=-ATEL(KJ,K1)
355 B(KJ+4*NC+MNM,K1+MNM)=ATEL(KJ,K1)
DO 358 KU=1,NC
358 B(KJ+4*NC+MNM,MM+NC+KU)=-BTEL(KJ,KU)
360 CCNTINUE
DO 370 J=1,NC
RC(J)=0.0
RQ(J+NC)=SAHAT(J)
RQ(J+2*NC)=SAHAT(J)
370 RC(J+3*NC)=ASHAT(J)
DO 375 I1=1,MNM
375 RQ(4*NC+I1)=Z(I1)
```

```

DO 380 IJ =1,NNC
380 RC(4*NC+MNM+IJ)=SBTEL(IJ)+ASTELE(IJ)
C
C SIGN INDICATES THE TYPE OF CONSTRAINTS.
C ONE FOR LESS THAN EQUAL TO TYPE OF CONSTRAINT.
C ZERO FOR EQUALITY CONSTRAINT.
C MINUS ONE FOR GREATER THAN EQUAL TYPE OF CONSTRAINT.
C
SIGN(M1)=0
MK=3*NC
DO 399 I=1,MK
399 SIGN(I)=1
KM1=MK+1
KM2=MK+NC+MNM
DO 420 I=KM1,KM2
420 SIGN(I)=0
KM3=KM2+1
KM4=KM2+NNC
DO 430 I=KM3,KM4
430 SIGN(I)=1
C
C ALL THE COEFFICIENT IN THE LP TABLEAU START LEAVING NGE-
C COLUMNS BLANK.
C
DC 600 I=1,M1
DC 600 J=1,N
600 B(I,NGE+J)=B(I,J)
GP=0
PG=0
C
CALL LINP (M1,N,NGE)
C
C CHECK IF SOLUTION IS UNBOUNDED OR INFEASIBLE THEN TERMINATE.
C

```

```

IF(GP.EQ.1) GO TO 1000
IF(PG.EQ.1) GO TO 1000
DO 308 J=1,KN
C
C REARRANGING THE INDICIES OF THE VARIABPES IN A SERIAL ORDER.
C
DO 308 I=2,KN
P(J)=KK(J)
IF(J-I)307,308,308
307 IF(P(J)-KK(I))308,308,299
299 KK(J)=KK(I)
KK(I)=P(J)
308 CCONTINUE
DO 298 I=1,KN
298 RQ(KK(I))=RRQ(KK(I))
C
C RBD IS RIGID BODY DISPLACEMENT.
C
IM=2*MNM
DO 290 I=1,IM
290 RBD1(I)=0.
C
C TO FIND THE RIGID BODY DISPLACEMENTS (SMALL-'Q')
C
DO 205 I=1,IM
IF(KK(I)-IM)199,199,205
199 KX=I
IF(KK(I).LE.2)GO TO 202
GO TO 275
202 IF(KK(I).EQ.2) GO TO 280
GO TO 285
280 RBD1(2)=-RQ(KK(I))
GO TO 205
285 RBD1(1)=RQ(KK(I))

```

```

GC TO 205
275 IF(IM.EQ.2) GO TO 205
282 IF(KK(I).EQ.3) GO TO 283
GO TO 284
283 RBD1(3)=RQ(KK(I))
GO TO 205
284 IF(KK(I).EQ.4)GO TO 279
GO TO 205
279 RBD1(4)=-RQ(KK(I))
205 CCNTINUE
N21=0
DO 281 J=1,MNM
RBD(J)=RBD1(J+N21)+RBD1(J+1+N21)
N21=N21+1
281 WRITE(6,214) N21,RBD(J)
214 FORMAT(/2X,'RBD(',I1,')=' ,E15.7/)
KX=KX+1
FX2=RQ(KK(KX))

```

C  
C  
C  
C

TO GET THE CONTOUR DESIGN VARIABLES.  
CCNTOUR DESIGN VARIABLES START WITH INDEX OF RQ AS 2\*MNM+2.

```

J1=NC+MM
DO 210 I=1,NC
J2=KX+1
JC=I+MM
IF(KK(J2)-J1)206,206,221
206 CONTINUE
IF(KK(J2).EQ.J0) GO TO 203
GO TO 221
203 KX=KX+1
SB(K(I))=RQ(KK(J2))
GO TO 210
221 SB(K(I))=0.0

```

```

210 CONTINUE
    WRITE(6,226) FX2
226 FCRMAT(/2X,'COST FN=',E15.7/)
    NCI=NC
    WRITE(6,211)
211 FORMAT(/1X,'SR.NO.',3X,'POINT',9X,'SHAT',15X,'EPHAT',15X,'SBHAT')
C
C   CONTACT STRESS VECTOR FORMATION.
C
    DO 224 I=1,NC
    JK=KX+1
    K12=MM+NC+I
    IF(KK(JK).EQ.K12) GO TO 277
    GO TO 278
277 KX=KX+1
    SHAT(I)=RQ(K12)
    GC TO 224
278 SHAT(I)=0.0
224 CCNTINUE
C
C   COMPUTE EPSLON 'HAT'.
C
    DO 225 I=1,NC
    EPHAT=SB(K(I))+ASHAT(I)
    DO 208 K8=1,MNM
208 EPHAT=EPHAT+AHAT(I,K8)*RBD(K8)
    DO 209 J=1,NC
209 EPHAT=EPHAT+BHAT(I,J)*SHAT(J)
225 WRITE(6,223) I,K(I),SHAT(I),EPHAT,SB(K(I))
223 FORMAT(/5X,I2,3X,I2,3X,E15.7,5X,E15.7,5X,E15.7)
C
C   COMPUTE EPSLON 'TELDA'.
C   CHECK WHICH OF THE EPSLON TELDA IS ZERO.
C

```

```

WRITE(6,235)
235 FCRMAT(/5X,'POINT',8X,'EPSLON TELDA'/)
DO 230 I=1,NNC
EPTL=SBTEL(I)+ASTEL(I)
DC 236 K7=1,MNM
236 EPTL=EPTL+ATEL(I,K7)*RBD(K7)
DC 240J=1,NC
240 EPTL=EPTL+BTEL(I,J)*SHAT(J)
WRITE(6,231) K11(I),EPTL
IF(EPTL.LE.1E-04) GO TO 232
GC TO 230
232 II(K11(I))=1
230 CCNTINUE
231 FCRMAT(/5X,I5,5X,E15.7)

```

```

C
C S-HAT IS THE VECTOR OF CONTACT FORCES.
C CHECK WHICH OF THE S-HAT ARE ZERO.
C POINTS WHERE S-HAT IS ZERO ,IS LIFTED.
C KNC ARE THE NO. OF PTS. LIFTED.
C

```

```

KNC=0
251 FCRMAT(/2X,'KNC-NO.OF PTS. WHICH CAN BE LIFTED=',I3)
DO 259 I=1,NC
IF(SHAT(I))259,249,259
249 KNC=KNC+1
II(K(I))=0
259 CCNTINUE
WRITE(6,251)KNC
NC=0
DO 270 I=1,NN
IF(II(I).EQ.0)GO TO 270
NC=NC+1
K11(NC)=I
270 CONTINUE

```

```

      NNC=NN-NC
      WRITE(6,255) NC,NNC
255  FORMAT(/2X,'NC=',I3,2X,'NNC=',I3/)
C
C   COMPARE THE CONTOUR BETWEEN THE TWO CONSECUTIVE ITRNS.
C
      IF(NCI-NC) 725,800,725
800  CONTINUE
      JIK=0
      DO 825 I=1,NC
      IF(K(I)-KI1(I)) 728,825,728
728  JIK=JIK+1
825  K(I)=KI1(I)
      IF(JIK.EQ.0) GO TO 700
      GO TO 725
700  WRITE(6,860)
860  FORMAT(/5X,'FINAL RESULTS'/)
      WRITE(6,875) FX2
875  FCRMAT(/1X,'COST FN=',F12.6/)
      WRITE(6,405)
405  FCRMAT(/20X,'INITIAL GAP'/)
      WRITE(6,2)(SSA(I),I=1,NN)
      WRITE(6,403)
403  FCRMAT(/20X,'TOLERANCE LIMIT'/)
      WRITE(6,2)(SA(I),I=1,NN)
      WRITE(6,880)
880  FCRMAT(/5X,'POINT',7X,'FINAL GAP ',7X,'CONTACT FORCE',6X
1 'CONTACT STRESS')
      DO 900 I=1,NC
      CST(I)=SHAT(I)/DEL
900  WRITE(6,890) K(I),SB(K(I)),SHAT(I),CST(I)
890  FORMAT(/5X,I5,5X,E15.7,5X,E15.7,5X,E15.7)
1000 CALL EXIT
      END

```

```

C *****
C *
C * SUBROUTINE LINEAR PROGRAMMING. *
C *
C *****
C
C SUBROUTINE LINP (M1,N,NGE)
C
C INTEGER RNM1,RNM2,CLNM1,CLNM2,BLNK ,LPFTN 1
1 IBN1(105),IBN2(105),NBN1(105),NBN2(105)
C
C INTEGER SIGN,GP,PG
C REAL PIVOT,LST,XNBP,FN,CJBAR,X,VALUE,BP(105),PI(105),XPI(105)
C REAL NBP
C
C COMMON B(105,105),RQ(105),NBP(105),SIGN(105),KK(105),ITRN
C COMMON RRQ(105),KN,GP,PG
C
C DATA BLNK/4H / LPFTN 5
C DATA NM1,NM2/'CCCC','AAAA'/
C
C INPUT PROGRAM LPFTN 8
C LPFTN 16
C LPFTN 17
C LPFTN 18
C
C NI = 5
C NC = 6
C IN=1
C M=M1-1
C DO 101 I=1,M
C IF(SIGN(I))108,107,106
106 BP(I)=0.
C GO TO 101
108 BP(I)=-1.0
C B(I,IN)=-1.0

```

```

      NBN1(IN)=NM2
      NBN2(IN)=IN
      NBP(IN)=0.
      IN=IN+1
      GO TO 101
107  BP(I)=-2.0
101  CCNTINUE
      DC 102 J=1,N
      NBN1(J+NGE)=NM1
102  NBN2(J+NGE)=J+NGE
      N=N+NGE
      DC 10 I=1,M
      IF(BP(I)+1.0) 19,11,12
11  IBN1(I)=BLNK
      IBN2(I)=BLNK
      GO TO 10
19  BP(I)=-1.0
      GO TO 11
12  CCNTINUE
      IBN1(I)=NM2
      IBN2(I)=I
10  CCNTINUE

```

```

C
C   ACCUMULATE COUNT OF INFEASIBILITIES
C

```

```

      NINF =0
      DO 6000 I=1,M
      IF(BP(I))6001,6000,6000
6001 NINF = NINF+1
6000 CCNTINUE

```

```

C
C   GENERATE INDICATORS FOR MINIMIZATION OF INFEASIBILITY
C
      DO 6101 J=1,N

```

```

LPFTN164
LPFTN165
LPFTN166
LPFTN167
LPFTN168
LPFTN169
LPFTN170

LPFTN172
LPFTN173
LPFTN174
LPFTN175

```

	XPI(J) = 0.				LPFTN176
	DO 6101 I=1,M				LPFTN177
	IF(BP(I))6102,6101,6101				LPFTN178
6102	XPI(J) = XPI(J)-B(I,J)				LPFTN179
6101	CCONTINUE				LPFTN180
	DC 6002 I=1,M				LPFTN181
6002	BP(I) = 0.				LPFTN182
	WRITE(6,401)				
401	FORMAT(/2X,'*****'/)				
	WRITE(6,400) ITRN				
400	FORMAT(/5X,'MAIN ITRN NO.=',I3/)				
	WRITE(6,402)				
402	FCRMT(/2X,'*****'/)				
	ITRN=ITRN+1				
	IPHASE = 1				LPFTN183
C					LPFTN184
C	MAIN ROUTINE				LPFTN185
	9201 WRITE(NO,9202)				
9202	FORMAT ('O ITERATION    VAR IN            VAR OUT            OBJ FN',/)				LPFTN188
	IT = 0				LPFTN189
54325	CCONTINUE				LPFTN190
C					LPFTN191
C	CALCULATE SHADOW PRICES				LPFTN192
C					LPFTN193
	DC 194 J=1,N				LPFTN194
	PI(J) = -NBP(J)				LPFTN195
	DC 194 I=1,M				LPFTN196
194	PI(J) = PI(J) + BP(I)*B(I,J)				LPFTN197
C					LPFTN198
C	SELECT BEST NONBASIS VECTOR				LPFTN199
C					LPFTN200
9101	LST = -.0000001				
	KCOL = 0				LPFTN202

	GO TO (751,552),IPHASE	LPFTN203
751	IF(NINF)54321,54321,552	LPFTN204
552	CONTINUE	LPFTN205
	DO 9102 J=1,N	LPFTN206
C		LPFTN207
C	IGNORE ARTIFICIAL VARIABLES	LPFTN208
C		LPFTN209
	IF(NBN1(J)-BLNK+NBN2(J)-BLNK)651,9102,651	LPFTN210
651	CCONTINUE	LPFTN211
	GO TO (6003,6004),IPHASE	LPFTN212
6003	IF(XPI(J)-LST)6005,6006,6006	LPFTN213
6005	KCOL=J	LPFTN214
	LST = XPI(J)	LPFTN215
	GO TO 9102	
6004	CCONTINUE	LPFTN217
	IF(PI(J)-LST)9103,9102,9102	LPFTN218
9103	KCOL = J	LPFTN219
	LST = PI(J)	LPFTN220
6006	CCONTINUE	LPFTN221
9102	CCONTINUE	LPFTN222
	IF (KCOL)54321,54321,9104	LPFTN223
C		LPFTN224
C	DETERMINE KEYROW	LPFTN225
C		LPFTN226
9104	KROW = 0	LPFTN227
	CJBAR = LST	LPFTN228
	LST = 1.0E20	LPFTN229
	DO 9105 I=1,M	LPFTN230
	IF(B(I,KCOL))9105,9105,9106	
9106	RATIO = RQ(I)/B(I,KCOL)	LPFTN232
	IF (RATIO-LST)9107,9105,9105	LPFTN233
9107	LST = RATIO	LPFTN234
	KROW=I	LPFTN235
9105	CONTINUE	LPFTN236

IF(KROW)9112,9112,9114	LPFTN237
9112 WRITE(NO,9113) NBN1(KCOL),NBN2(KCOL)	
9113 FORMAT(' VARIABLE ',A2,I3,' UNBOUNDED ')	
GP=GP+1	
GO TO 54323	LPFTN240
9114 CCNTINUE	LPFTN241
C	LPFTN242
C TRANSFORM	LPFTN243
C	LPFTN244
C DIVIDE BY PIVOT	LPFTN245
PIVOT = B(KROW,KCOL)	
DO 9108 J=1,N	LPFTN247
9108 B(KROW,J) = B(KROW,J)/PIVOT	LPFTN248
RQ(KROW) = RQ(KROW)/PIVOT	LPFTN249
DO 9109 I=1,M	LPFTN250
IF(I-KROW)9110,9109,9110	LPFTN251
9110 RC(I) = RQ(I) - RQ(KROW)*B(I,KCOL)	LPFTN252
DO 4444 J = 1, N	
IF(J-KCOL)9111,4444,9111	LPFTN254
9111 B(I,J) = B(I,J) - B(KROW,J)*B(I,KCOL)	LPFTN255
4444 CCNTINUE	
9109 CCNTINUE	LPFTN256
DO 9300 I=1,M	LPFTN257
9300 B(I,KCOL) = -B(I,KCOL)/PIVOT	LPFTN258
B(KROW,KCOL) = 1.0/PIVOT	LPFTN259
C	LPFTN260
C INTERCHANGE BASIS AND NONBASIS VARIABLES	LPFTN2
C	LPFTN262
RNM1 = NBN1(KCOL)	LPFTN263
RNM2 = NBN2(KCOL)	LPFTN264
NBN1(KCOL) = IBN1(KROW)	LPFTN265
NBN2(KCOL) = IBN2(KROW)	LPFTN266
IBN1(KROW) = RNM1	LPFTN267
IBN2(KROW) = RNM2	LPFTN268

LST = NBP(KCOL)	LPFTN269
NBP(KCOL) = BP(KROW)	LPFTN270
BP(KROW) = LST	LPFTN271
IT = IT + 1	LPFTN272
IF(NBN1(KCOL)-BLNK+NBN2(KCOL)-BLNK) 6201,6200,6201	LPFTN273
6200 NINF = NINF-1	LPFTN274
6201 CCNTINUE	LPFTN275
C	
C     COMPUTE OBJECTIVE FUNCTION	LPFTN277
C	LPFTN278
FN = 0.	LPFTN279
DC 9301 I=1,M	LPFTN280
9301 FN = FN + BP(I)*RQ(I)	LPFTN281
GC TO (7000,7001),IPHASE	LPFTN282
7000 SAVE = PI(KCOL)	LPFTN283
DO 7003 J=1,N	LPFTN284
PI(J) = PI(J) - SAVE*B(KROW,J)	LPFTN285
XPI(J) = XPI(J) - CJBAR*B(KROW,J)	LPFTN286
7003 CCNTINUE	LPFTN287
PI(KCOL) = -SAVE/PIVOT	LPFTN288
XPI(KCOL) = -CJBAR/PIVOT	LPFTN289
GC TO 7004	LPFTN290
7001 CCNTINUE	
DC 9302 J=1,N	LPFTN292
9302 PI(J) = PI(J) - CJBAR*B(KROW,J)	LPFTN293
PI(KCOL) = -CJBAR/PIVOT	LPFTN294
7004 CCNTINUE	LPFTN295
C     CHECK FOR ESSENTIAL ZERO	LPFTN296
DO 6111 I=1,M	LPFTN297
DO 6111 J=1,N	LPFTN298
X=B(I,J)	LPFTN299
IF(ABS(X)-.0000001)6112,6112,6111	LPFTN300
6112 B(I,J) = 0.	LPFTN301
6111 CONTINUE	LPFTN302

C		LPFTN303
C		LPFTN304
C	LOG ITERATION	LPFTN305
C		
	WRITE(NO,9120) IT, IBN1(KROW), IBN2(KROW), NBN1(KCOL), NBN2(KCOL), FN	
9120	FORMAT(I9,7X,A2,I3,8X,A2,I3,3X,F13.3)	
	GO TO 9101	LPFTN309
C		LPFTN310
C		LPFTN311
54321	CCONTINUE	LPFTN312
	IF(IPHASE-1)8000,8000,54322	LPFTN313
8000	IPHASE = 2	LPFTN314
	IF(NINF)8003,8003,8004	LPFTN315
8004	WRITE(NO,8005)	
8005	FCRMT('O SOLUTION INFEASIBLE',/)	LPFTN317
	PG=PG+1	
	GO TO 54323	
8003	CCONTINUE	LPFTN319
	WRITE(NO,8002)	
8002	FCRMT('O SOLUTION FEASIBLE ',/)	LPFTN321
	GO TO 54325	
54322	CONTINUE	LPFTN323
C		LPFTN324
C	OUTPUT ROUTINE	LPFTN325
C		LPFTN326
	WRITE(NO,301) IT, FN	
301	FCRMT('1',' ITERATION',I5,' OBJ FN ',F15.3/)	LPFTN328
	WRITE(NO,302)	
302	FORMAT(3X,'BASIS VAR',17X,'AMOUNT')	
	KN=0	
	DO 3033 I=1,M	LPFTN332
C		LPFTN333
C	COST RANGING	LPFTN334
C		LPFTN335

	VALUE = 1.0E20	LPFTN336
	LST = 1.0E20	LPFTN337
	DC 12300 J=1,N	LPFTN338
	IF(NBN1(J)-BLNK+NBN2(J)-BLNK)12305,12300,12305	LPFTN339
12305	CCONTINUE	LPFTN340
	IF(B(I,J))12301,12300,12302	LPFTN341
12302	X=PI(J)/B(I,J)	LPFTN342
	IF(X-LST)12303,12300,12300	LPFTN343
12303	LST=X	LPFTN344
	GO TO 12300	LPFTN345
12301	X=-PI(J)/B(I,J)	LPFTN346
	IF(X-VALUE)12304,12300,12300	LPFTN347
12304	VALUE = X	LPFTN348
12300	CCONTINUE	LPFTN349
	LST = BP(I) - LST	LPFTN350
	VALUE = BP(I) + VALUE	LPFTN351
C		
C	VARIABLES WITH NAMES AS 'CC' ARE SEPERATED .	
C	THESE VARIABLES ARE THE DESIGN VARIABLES OF LP PROBLEM.	
C		
C		
C	VARIABLES WITH NAMES 'AA' ARE ELIMINATED IN THE NEXT STEP.	
C	THESE ARE THE SLACK VARIABLES.	
C		
	IF(IBN1(I)-NM1)3033,306,3033	
306	KN=KN+1	
	KK(KN)=IBN2(I)	
	RRQ(KK(KN))=RQ(I)	
	WRITE(6,304)IBN1(I),IBN2(I),RQ(I)	
3033	CCONTINUE	
304	FORMAT(7X,A2,I3,7X,F16.6)	
54323	CCONTINUE	LPFTN 19
	RETURN	
	END	LPFTN365

```

C *****
C *
C * INPUT DATA. *
C *
C *****

```

```

      P          U          AL          DEL          MNM  NN   NC   NNC  NGE
1000.    0.01    1.44    0.06           1   25  12  13   0

```

A1-MATRIX.

```

0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01
0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01
0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01
0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01
0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01  0.1000000E 01

```

T-MATRIX.

ROW 1

```

0.5000000E-04  0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00
0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00
0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00
0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00
0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00  0.0000000E 00

```

ROW 2

```

0.0000000E 00  0.5002033E-04  0.5082351E-07  0.8131758E-07  0.1118117E-06
0.1423058E-06  0.1728000E-06  0.2032941E-06  0.2337882E-06  0.2642823E-06
0.2947765E-06  0.3252706E-06  0.3557647E-06  0.3862589E-06  0.4167530E-06
0.4472471E-06  0.4777413E-06  0.5082351E-06  0.5387294E-06  0.5692230E-06

```

0.5997176E-06 0.6302112E-06 0.6607058E-06 0.6911994E-06 0.7216940E-06

ROW 3

0.0000000E 00 0.5082351E-07 0.5016263E-04 0.2846118E-06 0.4065882E-06  
0.5285647E-06 0.6505411E-06 0.7725175E-06 0.8944938E-06 0.1016469E-05  
0.1138446E-05 0.1260422E-05 0.1382399E-05 0.1504375E-05 0.1626352E-05  
0.1748329E-05 0.1870305E-05 0.1992281E-05 0.2114258E-05 0.2236233E-05  
0.2358211E-05 0.2480186E-05 0.2602163E-05 0.2724139E-05 0.2846116E-05

ROW 4

0.0000000E 00 0.8131758E-07 0.2846118E-06 0.5054889E-04 0.8233413E-06  
0.1097787E-05 0.1372235E-05 0.1646682E-05 0.1921129E-05 0.2195577E-05  
0.2470023E-05 0.2744470E-05 0.3018918E-05 0.3293365E-05 0.3567811E-05  
0.3842259E-05 0.4116706E-05 0.4391150E-05 0.4665600E-05 0.4940044E-05  
0.5214493E-05 0.5488939E-05 0.5763387E-05 0.6037832E-05 0.6312281E-05

ROW 5

0.0000000E 00 0.1118117E-06 0.4065882E-06 0.8233413E-06 0.5130107E-04  
0.1788988E-05 0.2276894E-05 0.2764800E-05 0.3252706E-05 0.3740612E-05  
0.4228518E-05 0.4716424E-05 0.5204330E-05 0.5692236E-05 0.6180142E-05  
0.6668048E-05 0.7155954E-05 0.7643855E-05 0.8131765E-05 0.8619662E-05  
0.9107576E-05 0.9595473E-05 0.1008339E-04 0.1057128E-04 0.1105920E-04

ROW 6

0.0000000E 00 0.1423058E-06 0.5285647E-06 0.1097787E-05 0.1788988E-05  
0.5254117E-04 0.3303528E-05 0.4065882E-05 0.4828235E-05 0.5590588E-05  
0.6352941E-05 0.7115294E-05 0.7877644E-05 0.8639997E-05 0.9402351E-05  
0.1016470E-04 0.1092706E-04 0.1168940E-04 0.1245176E-04 0.1321410E-04  
0.1397646E-04 0.1473880E-04 0.1550115E-04 0.1626350E-04 0.1702584E-04

ROW 7

0.0000000E 00	0.1728000E-06	0.6505411E-06	0.1372235E-05	0.2276894E-05
0.3303528E-05	0.5439114E-04	0.5488941E-05	0.6586730E-05	0.7684514E-05
0.8782305E-05	0.9880087E-05	0.1097788E-04	0.1207567E-04	0.1317344E-04
0.1427124E-04	0.1536903E-04	0.1646680E-04	0.1756460E-04	0.1866238E-04
0.1976016E-04	0.2085796E-04	0.2195574E-04	0.2305352E-04	0.2415132E-04

ROW 8

0.0000000E 00	0.2032941E-06	0.7725175E-06	0.1646682E-05	0.2764800E-05
0.4065882E-05	0.5488941E-05	0.5697299E-04	0.8467199E-05	0.9961411E-05
0.1145562E-04	0.1294982E-04	0.1444404E-04	0.1593825E-04	0.1743247E-04
0.1892669E-04	0.2042089E-04	0.2191508E-04	0.2340930E-04	0.2490351E-04
0.2639773E-04	0.2789193E-04	0.2938615E-04	0.3088034E-04	0.3237458E-04

ROW 9

0.0000000E 00	0.2337882E-06	0.8944938E-06	0.1921129E-05	0.3252706E-05
0.4828235E-05	0.6586730E-05	0.8467199E-05	0.6040865E-04	0.1236027E-04
0.1431190E-04	0.1626351E-04	0.1821514E-04	0.2016676E-04	0.2211839E-04
0.2407002E-04	0.2602163E-04	0.2797325E-04	0.2992488E-04	0.3187648E-04
0.3382812E-04	0.3577973E-04	0.3773137E-04	0.3968297E-04	0.4163462E-04

ROW 10

0.0000000E 00	0.2642823E-06	0.1016469E-05	0.2195577E-05	0.3740612E-05
0.5590588E-05	0.7684514E-05	0.9961411E-05	0.1236027E-04	0.6482012E-04
0.1729015E-04	0.1976016E-04	0.2223020E-04	0.2470022E-04	0.2717024E-04
0.2964027E-04	0.3211029E-04	0.3458030E-04	0.3705033E-04	0.3952034E-04
0.4199038E-04	0.4446038E-04	0.4693042E-04	0.4940042E-04	0.5187046E-04

ROW 11

0.0000000E 0C	0.2947765E-06	0.1138446E-05	0.2470023E-05	0.4228518E-05
0.6352941E-05	0.8782305E-05	0.1145562E-04	0.1431190E-04	0.1729015E-04
0.7032939E-04	0.2337880E-04	0.2642823E-04	0.2947764E-04	0.3252705E-04
0.3557646E-04	0.3862588E-04	0.4167526E-04	0.4472470E-04	0.4777408E-04
0.5082351E-04	0.5387289E-04	0.5692233E-04	0.5997172E-04	0.6302114E-04

ROW 12

0.0000000E 00	0.3252706E-06	0.1260422E-05	0.2744470E-05	0.4716424E-05
0.7115294E-05	0.9880087E-05	0.1294982E-04	0.1626351E-04	0.1976016E-04
0.2337880E-04	0.7705843E-04	0.3074821E-04	0.3443801E-04	0.3812779E-04
0.4181759E-04	0.4550737E-04	0.4919713E-04	0.5288694E-04	0.5657670E-04
0.6026651E-04	0.6395628E-04	0.6764609E-04	0.7133585E-04	0.7502566E-04

ROW 13

0.0000000E 00	0.3557647E-06	0.1382399E-05	0.3018918E-05	0.5204330E-05
0.7877644E-05	0.1097788E-04	0.1444404E-04	0.1821514E-04	0.2223020E-04
0.2642823E-04	0.3074821E-04	0.8512923E-04	0.3952038E-04	0.4391152E-04
0.4830268E-04	0.5269384E-04	0.5708495E-04	0.6147614E-04	0.6586725E-04
0.7025844E-04	0.7464956E-04	0.7904074E-04	0.8343186E-04	0.8782303E-04

ROW 14

0.0000000E 00	0.3862589E-06	0.1504375E-05	0.3293365E-05	0.5692236E-05
0.8639997E-05	0.1207567E-04	0.1593825E-04	0.2016676E-04	0.2470022E-04
0.2947764E-04	0.3443801E-04	0.3952038E-04	0.9466373E-04	0.4981723E-04
0.5497073E-04	0.6012425E-04	0.6527771E-04	0.7043124E-04	0.7558471E-04
0.8073825E-04	0.8589170E-04	0.9104525E-04	0.9619871E-04	0.1013523E-03

ROW 15

0.0000000E 00	0.4167530E-06	0.1626352E-05	0.3567811E-05	0.6180142E-05
0.9402351E-05	0.1317344E-04	0.1743247E-04	0.2211839E-04	0.2717024E-04

0.3252705E-04	0.3812779E-04	0.4391152E-04	0.4981723E-04	0.1057839E-03
0.6176077E-04	0.6773762E-04	0.7371441E-04	0.7969131E-04	0.8566810E-04
0.9164499E-04	0.9762179E-04	0.1035987E-03	0.1095755E-03	0.1155524E-03

ROW 16

0.0000000E 00	0.4472471E-06	0.1748329E-05	0.3842259E-05	0.6668048E-05
0.1016470E-04	0.1427124E-04	0.1892669E-04	0.2407002E-04	0.2964027E-04
0.3557646E-04	0.4181759E-04	0.4830268E-04	0.5497073E-04	0.6176077E-04
0.1186118E-03	0.7547297E-04	0.8233408E-04	0.8919531E-04	0.9605644E-04
0.1029176E-03	0.1097788E-03	0.1166400E-03	0.1235010E-03	0.1303622E-03

ROW 17

0.0000000E 00	0.4777413E-06	0.1870305E-05	0.4116706E-05	0.7155954E-05
0.1092706E-04	0.1536903E-04	0.2042089E-04	0.2602163E-04	0.3211029E-04
0.3862588E-04	0.4550737E-04	0.5269384E-04	0.6012425E-04	0.6773762E-04
0.7547297E-04	0.1332693E-03	0.9107571E-04	0.9888226E-04	0.1066886E-03
0.1144952E-03	0.1223016E-03	0.1301082E-03	0.1379146E-03	0.1457212E-03

ROW 18

0.0000000E 00	0.5082351E-06	0.1992281E-05	0.4391150E-05	0.7643855E-05
0.1168940E-04	0.1646680E-04	0.2191508E-04	0.2797325E-04	0.3458030E-04
0.4167526E-04	0.4919713E-04	0.5708495E-04	0.6527771E-04	0.7371441E-04
0.8233408E-04	0.9107571E-04	0.1498782E-03	0.1086910E-03	0.1175037E-03
0.1263166E-03	0.1351293E-03	0.1439421E-03	0.1527549E-03	0.1615677E-03

ROW 19

0.0000000E 00	0.5387294E-06	0.2114258E-05	0.4665600E-05	0.8131765E-05
0.1245176E-04	0.1756460E-04	0.2340930E-04	0.2992488E-04	0.3705033E-04
0.4472470E-04	0.5288694E-04	0.6147614E-04	0.7043124E-04	0.7969131E-04
0.8919531E-04	0.9888226E-04	0.1086910E-03	0.1685610E-03	0.1284410E-03

0.1383211E-03 0.1482012E-03 0.1580813E-03 0.1679613E-03 0.1778415E-03

ROW 20

0.0000000E 00 0.5692230E-06 0.2236233E-05 0.4940044E-05 0.8619662E-05  
0.1321410E-04 0.1866238E-04 0.2490351E-04 0.3187648E-04 0.3952034E-04  
0.4777408E-04 0.5657670E-04 0.6586725E-04 0.7558471E-04 0.8566810E-04  
0.9605644E-04 0.1066886E-03 0.1175037E-03 0.1284410E-03 0.1894390E-03  
0.1504476E-03 0.1614558E-03 0.1724642E-03 0.1834724E-03 0.1944808E-03

ROW 21

0.0000000E 00 0.5997176E-06 0.2358211E-05 0.5214493E-05 0.9107576E-05  
0.1397646E-04 0.1976016E-04 0.2639773E-04 0.3382812E-04 0.4199038E-04  
0.5082351E-04 0.6026651E-04 0.7025844E-04 0.8073825E-04 0.9164499E-04  
0.1029176E-03 0.1144952E-03 0.1263166E-03 0.1383211E-03 0.1504476E-03  
0.2126351E-03 0.1748324E-03 0.1870302E-03 0.1992277E-03 0.2114255E-03

ROW 22

0.0000000E 00 0.6302112E-06 0.2480186E-05 0.5488939E-05 0.9595473E-05  
0.1473880E-04 0.2085796E-04 0.2789193E-04 0.3577973E-04 0.4446038E-04  
0.5387289E-04 0.6395628E-04 0.7464956E-04 0.8589170E-04 0.9762179E-04  
0.1097788E-03 0.1223016E-03 0.1351293E-03 0.1482012E-03 0.1614558E-03  
0.1748324E-03 0.2382702E-03 0.2017181E-03 0.2151659E-03 0.2286140E-03

ROW 23

0.0000000E 00 0.6607058E-06 0.2602163E-05 0.5763387E-05 0.1008339E-04  
0.1550115E-04 0.2195574E-04 0.2938615E-04 0.3773137E-04 0.4693042E-04  
0.5692233E-04 0.6764609E-04 0.7904074E-04 0.9104525E-04 0.1035987E-03  
0.1166400E-03 0.1301082E-03 0.1439421E-03 0.1580813E-03 0.1724642E-03  
0.1870302E-03 0.2017181E-03 0.2664672E-03 0.2312262E-03 0.2459851E-03

ROW 24

0.0000000E 00	0.6911994E-06	0.2724139E-05	0.6037832E-05	0.1057128E-04
0.1626350E-04	0.2305352E-04	0.3088034E-04	0.3968297E-04	0.4940042E-04
0.5997172E-04	0.7133585E-04	0.8343186E-04	0.9619871E-04	0.1095755E-03
0.1235010E-03	0.1379146E-03	0.1527549E-03	0.1679613E-03	0.1834724E-03
0.1992277E-03	0.2151659E-03	0.2312262E-03	0.2973471E-03	0.2634786E-03

ROW 25

0.0000000E 00	0.7216940E-06	0.2846116E-05	0.6312281E-05	0.1105920E-04
0.1702584E-04	0.2415132E-04	0.3237458E-04	0.4163462E-04	0.5187046E-04
0.6302114E-04	0.7502566E-04	0.8782303E-04	0.1013523E-03	0.1155524E-03
0.1303622E-03	0.1457212E-03	0.1615677E-03	0.1778415E-03	0.1944808E-03
0.2114255E-03	0.2286140E-03	0.2459851E-03	0.2634786E-03	0.3310333E-03

POINTS IN CONTACT OBTAINED FROM QUADRATIC PROGRAMMING.

1 2 3 4 5 6 7 8 9 10 11 12

