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# **Evaluation Tests of Select Fuel Additives for Potential Use in U.S. Army Corps of Engineers Diesel Engines**

Michael Tubman and Timothy Welp

July 2016



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# **Evaluation Tests of Select Fuel Additives for Potential Use in U.S. Army Corps of Engineers Diesel Engines**

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## Abstract

The U.S. Army Corps of Engineers (USACE) has approximately 2,300 floating plant assets that consist of barges, tow boats, floating cranes, survey boats, patrol boats, and fleet dredges that have diesel engines. Under the Dredging Operations and Environmental Research (DOER) program, diesel fuel additives were tested to evaluate their potential for reducing diesel fuel consumption and cost.

Four fuel additives were tested to evaluate their potential for reducing diesel fuel consumption and cost:

- An ethanol injection system
- Envirofuels Diesel Fuel Catalyst
- DurAlt Fuel Conditioner
- Lucas Fuel Treatment.

Fuel usage was measured while using the additives with diesel fuel (candidate tests) and compared to fuel usage under the same conditions while using only standard diesel fuel (baseline tests). The evaluations were conducted in the field under actual, in-use operating conditions. The results are applicable to the host engines and operating conditions, but similar results can be expected for similar engine families, year of manufacture, and operating regimes. While the Envirofuels, DurAlt, and Lucas additives showed limited fuel reduction in select operation conditions, only the ethanol injection system consistently showed potential to reduce diesel fuel consumption, which may be due to its higher injection volume.

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# Contents

<b>Abstract .....</b>	<b>ii</b>
<b>Figures and Tables.....</b>	<b>iv</b>
<b>Preface.....</b>	<b>vi</b>
<b>Acknowledgement.....</b>	<b>vii</b>
<b>Unit Conversion Factors .....</b>	<b>viii</b>
<b>1 Introduction.....</b>	<b>1</b>
Background .....	1
Objective .....	1
Approach.....	2
<b>2 Test Sites and Engines.....</b>	<b>3</b>
<b>3 Test Procedures and Equipment.....</b>	<b>5</b>
<b>4 Fuel Additives.....</b>	<b>8</b>
<b>5 Test Results.....</b>	<b>11</b>
Lock and Dam No. 4 .....	11
Taggatz.....	15
Pathfinder.....	22
<b>6 Summary and Conclusions .....</b>	<b>25</b>
<b>References .....</b>	<b>27</b>
<b>Report Documentation Page</b>	

# Figures and Tables

## Figures

Figure 1. The quarters boat <i>Taggatz</i> .....	3
Figure 2. The towboat <i>Pathfinder</i> .....	4
Figure 3. Lock and Dam No. 4 load-bank test results, 50% load.....	12
Figure 4. Lock and Dam No. 4 load-bank test results, 75% load.....	12
Figure 5. Lock and Dam No. 4 load-bank test results, 90% load.....	13
Figure 6. Lock and Dam No. 4 BTU input and engine output.....	14
Figure 7. <i>Taggatz</i> port generator baseline and candidate BSFC, 50% load.....	17
Figure 8. <i>Taggatz</i> port generator baseline and candidate BSFC, 75% load.....	17
Figure 9. <i>Taggatz</i> port generator baseline and candidate BSFC, 85% load.....	18
Figure 10. <i>Taggatz</i> starboard generator and candidate baseline BSFC, 25% load.....	19
Figure 11. <i>Taggatz</i> starboard generator baseline BSFC, 50% load.....	20
Figure 12. <i>Taggatz</i> starboard generator baseline BSFC, 75% load.....	20
Figure 13. <i>Taggatz</i> starboard generator baseline BSFC, 85% load.....	21
Figure 14. <i>Pathfinder</i> bollard push test results at different loads.....	22
Figure 15. Duty-cycle test results at different loads.....	24

## Tables

Table 1. Ethanol properties.....	8
Table 2. Additives (DFC) and fuel amounts during the degeening period.....	9
Table 3. DFC properties.....	9
Table 4. DurAlt Fuel Conditioner properties.....	10
Table 5. Lucas Fuel Treatment additive properties.....	10
Table 6. Additives and fuel for break-in periods and testing.....	10
Table 7. Lock and Dam No. 4 load-bank test results, 50% load.....	11
Table 8. Lock and Dam No. 4 load-bank test results, 75% load.....	11
Table 9. Lock and Dam No. 4 load-bank test results, 90% load.....	12
Table 10. Lock and Dam No. 4 estimated CO <sub>2</sub> emission differences.....	14
Table 11. <i>Taggatz</i> port test loads.....	15
Table 12. <i>Taggatz</i> starboard test loads.....	15
Table 13. <i>Taggatz</i> port test results, 50% load.....	16
Table 14. <i>Taggatz</i> port test results, 75% load.....	16
Table 15. <i>Taggatz</i> port test results, 85% load.....	16
Table 16. <i>Taggatz</i> starboard test results, 25% load.....	18
Table 17. <i>Taggatz</i> starboard test results, 50% load.....	18
Table 18. <i>Taggatz</i> starboard test results, 75% load.....	19

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Table 19. *Taggatz* starboard test results, 85% load. ....19

Table 20. *Pathfinder* bollard-push baseline and candidate fuel consumption changes. ....23

Table 21. *Pathfinder* duty-cycle baseline and candidate fuel consumption changes. ....23

## Preface

This study was conducted for the Headquarters, U.S. Army Corps of Engineers (HQUSACE) under the Dredging Operations and Environmental Research (DOER) Program, Work Unit 456009, “Feasibility of Using Biodiesel Additives.” The technical monitor was Dr. Todd Bridges (CEERD-EM-D).

The work was performed by the Coastal Engineering Branch of the Navigation Division (CEERD-N), U.S. Army Engineer Research and Development Center, Coastal and Hydraulics Laboratory (ERDC-CHL). At the time of publication, Tanya Beck was Chief, CEERD-HN-C; Dr. Jackie Pettway was Chief, CEERD-HN; and Jeffery Lillycrop, CEERD-HT, was the Technical Director for Navigation. The Director of ERDC-CHL was José E. Sánchez.

This effort was supported by the by USACE Dredging Operations and Environmental Research (DOER) program managed at the U.S. Army Engineer Research and Development Center (ERDC) by Dr. Todd Bridges, Environmental Laboratory (EL). The study was performed in conjunction with the Southern Research Institute, Durham, NC. Southern personnel included Tim Hansen (Green House Gas Center Director), Bill Chatterton (Project Manager), William Crews (Senior Project Leader), and Eric Ringler (Quality Assurance Manager). USACE staff managed all vessel operations and scheduling, managed the purchase and implementation of the fuel additives in accordance with manufacturer specifications, and managed the consistent fuel supplies for the test period. Southern Research Institute managed the test campaign, including test strategy development and documentation, coordination and execution of all field testing, and data validation, analysis, quality assurance and quality control (QA/QC), and reporting.

At the time of publication of this report, the Commander of ERDC was COL Bryan S. Green, and the Director was Dr. Jeffery P. Holland.

## Acknowledgement

This report was sponsored by the USACE DOER program (<http://el.erd.c.usace.army.mil/dots/doer/doer.html>). This study would not have been possible without the participation, innovation, and dedication of personnel from the U.S Army Engineer Districts St. Louis (MVS) and St. Paul (MVP). MVS personnel included Lance Engle (Project Manager) and crew of the *Pathfinder*: Terry Bequette (Skipper), Brett Leavitt, Larry Baltzell (Nub), Mike Morgan (Spike), and Tom Brace. MVP personnel at the Fountain City Service Base included Greg Frankosky (Chief Physical Support Branch), James Maybach (Civil Engineer), Jake Bernhardt (Chief Engineer), and Bobbie Davis (Operator Equipment Repairman, Lock and Dam 4).

## Unit Conversion Factors

Non-SI units of measurement in this report can be converted to SI (metric) units as follows:

Multiply	By	To Obtain
feet	0.3048	meters
pound-mass per cubic foot	16.0185	grams per cubic centimeter
pound-force per square inch	47.8803	Pascals
Grams per cubic centimeter (g/cm <sup>3</sup> ) can be converted to grams per liter (g/L) by multiplying by 1000.		

# 1 Introduction

## Background

The U.S. Army Corps of Engineers (USACE) has approximately 2,300 floating plant assets that consist of barges, tow boats, floating cranes, survey boats, patrol boats, and fleet dredges that have diesel engines. The fiscal year 2010 diesel fuel consumption for the floating plant assets was approximately 8.29 million gallons. In addition, there are other uses of diesel engines within the USACE. Providing backup generator power at locks and dams is an example. On 5 October 2009, Executive Order 13514 was issued by President Obama (Office of the Press Secretary 2009) that requires Federal agencies to develop a strategic sustainability performance plan (SSPP) to reduce energy consumption and greenhouse gas emissions, increase agency use of renewable energy, and reduce the use of fossil fuels. For USACE floating plant, one of the main strategies of the USACE SSPP is reducing diesel fuel consumption, and a potential way to accomplish that is to use additives that are proven to reduce regular petroleum-derived diesel consumption.

Under the Dredging Operations and Environmental Research (DOER) program, the following diesel fuel additives were tested to evaluate their potential for reducing diesel fuel consumption (and reducing reliance on fossil fuels and cost:

- An ethanol injection system
- Envirofuels Diesel Fuel Catalyst
- DurAlt Fuel Conditioner
- Lucas Fuel Treatment.

These four technologies were selected from candidates proposed by various districts and subsequently tested on some of the diesel plant of those respective districts.

## Objective

The objective of this study was to evaluate various fuel additives to determine their potential for reducing diesel fuel consumption.

## Approach

Performance evaluations were conducted on diesel generator sets at the USACE Lock and Dam No. 4 Service Base in Alma, WI, the quarters boat *Taggatz* located near Wabasha, MN, and the towboat *Pathfinder* located in Saint Louis, MO. The evaluations were conducted in the field under actual, in-use operating conditions. The results are applicable to the host engines and operating conditions, but similar results can be expected for similar engine families, year of manufacture, and operating regimes.

Development of the test strategy, coordination, and execution of the field testing, data validation, data analysis, quality assurance and control, and reporting was managed by Southern Research Institute's Advanced Energy and Transportation Technologies Department, Durham, NC (referred to hereafter as Southern). Much of this report was compiled from Southern's test reports (Southern 2009, 2010). The testing concept and approach were based partly on *Generic In-Use Protocol for Non-road Equipment* (Richards and Haggis 2007) developed by the Southern Research Institute for the New York State Energy Research and Development Authority. The generic protocol provides overall test campaign designs, procedures for developing duty cycles, instrument specifications, step-by-step test procedures, and analytical techniques. Site-specific protocols were written to provide information about the individual test sites, nonroad diesel equipment and other details unique to the testing.

The general procedure for calculating fuel consumption was the following:

1. Download the data in Excel format.
2. Subtract the fuel return flow from the fuel supply flow.
3. Calculate the average pounds per hour fuel usage for each run period.
4. Chart the fuel usage throughout the test period and look for abnormalities (such as spikes caused by air bubbles in the meters).
5. Download the torque data in Excel format and sum the torque from each shaft.
6. Time-align the power/torque data with the fuel data.
7. Perform calculations of averages and standard deviations over the test period.
8. Compute the mean gal/bhp-hr over the test.
9. Determine the difference between the baseline test and the candidate (additive) test.
10. Determine the statistical significance of the differences between the baseline and additive performance and calculate the 95% confidence interval on the difference.

## 2 Test Sites and Engines

Lock and Dam No. 4 is located near Alma, WI, and Kellogg, MN, on the upper Mississippi River, positioned approximately at river mile 752.8. It was constructed and placed in operation in May 1935. Its last major rehabilitation was from 1988 to 1994. The dam consists of a concrete structure 1367 ft long with six roller gates, 22 retainer gates, and an earth embankment 5400 ft long. The lock is 110 ft wide by 600 ft long.

The alternative power system at the lock uses a Cummins NTA855-G2 inline 6-cylinder engine with a rated brake horsepower (bhp) of 466 at 1800 revolutions per minute (rpm) and a displacement of 855 cu in. Manufactured in 1990, it has only 400 hr of service. It is connected to an Onan Generator, Model #300DFCB36975E with a 350 kW output.

The *Taggatz* is a 160 (length) x 40 (beam) x 11.5 (draft) ft quarters boat (Figure 1). It operates in the upper Mississippi River out of the St. Paul, MN, USACE service base. The engines for the testing are part of the two generator sets onboard. They are Komatsu SA6D140-1 engines rated 500 bhp at 1800 rpm and displacements of 855 cu in. They were at early life with just under 2000 hr of use. Both engines were coupled to Northern Lights generators, Model #M6140AL2-330KW with 330 kW output.

Figure 1. The quarters boat *Taggatz*.



The *Pathfinder* is a 75 (length) x 30 (beam) x 8.5 (draft) ft twin-screw towboat that displaces 210 tons (Figure 2). It also operates in the upper Mississippi River out of the St. Paul, MN, USACE service base. It is powered by two Caterpillar 3412CDITA engines rated 671 bhp at 1800 rpm. The use on the port engine was 3,302 hr since a rebuild in 2007, and 10,696 hr on the starboard engine since a 2006 rebuild. After the port engine rebuild, the dynamometer specifications indicated that it

would produce a maximum of 664 bhp at 1845 rpm. Both engines have after-coolers and water-cooled turbochargers and are coupled to twin-disc MG-520 transmission drives with 5:1 gear reduction ratios. The shifting is controlled by a pneumatic system manufactured by WABCO Logic Master, which also links the pilot control house to the regulator and the main engine throttles and the clutch. The propeller shafts are connected to the rear of the transmission and travel through the hull and are supported by Johnson Duramax bearings. The shafts are made of Aquamet 15 stainless steel and are 5 in. in diameter and 20.75 ft in length.

Figure 2. The towboat *Pathfinder*.



### 3 Test Procedures and Equipment

*Duty cycles* are detailed descriptions of equipment maneuvers observed during testing. Equipment maneuvers may be described as individual events. For the engines coupled to the generators, events are idle, low, mid-range, and high generator loading. For the *Pathfinder*, events are backing, travel forward, turning, docking, etc. Composite events consist of a combination of individual events over varying time periods. The *Pathfinder*, for example, may combine a multiple of simple events such as slight forward travel with a reverse load on the other engine to turn while maintaining both engines at a constant speed for long intervals. A *simple* duty cycle is an arbitrary arrangement of single or composite events of specified duration (from 15 minutes to 1 hr to allow a reasonable number of test runs during a typical day) under controlled conditions that is representative of a typical work activity task. Simple cycles need to be repeatable as determined by the appropriate cycle criteria.

For the Lock and Dam No. 4 and the *Taggatz* engine-generator sets, simple duty-cycle testing consisted of a series of specified-generator kilowatt loads with the diesel engines running at 1800 rpm. The design generator loads were 25%, 50%, 75%, and 90% for the Lock and Dam No. 4 generator and 25%, 50%, 75%, and 85% for the *Taggatz* generators. The tests at each load were designed to last 20 minutes after a stabilization period. However, due to cooling restrictions on the *Taggatz* at the higher loads, the testing was shortened to 7 minutes for the 75% load and 3 minutes for the 85% load.

Simple duty-cycle testing for the *Pathfinder* was based on the “Guidelines for Bollard Pull Test Procedure” (American Bureau of Shipping 2006) and *Surface Vehicle Recommended Practice, Joint TMC/SAE Fuel Consumption Test Procedure – Type II* (SAE International 1986). The bollard-push duty cycles test design specified engine powers of idle (engine speed of 630 rpm with reduction gear in forward-shaft speed, 125 rpm), 25% (1132 rpm), 50% (1425 rpm), 75% (1630 rpm), and 100% while pushing against a rock wall. It was found that the *Pathfinder* starboard engine could not be operated at 100% due to propeller cavitations, so the test design was modified to load the engines at 79.4% (1660 rpm) in place of the 100% load test. The push tests were conducted at (closed) Lock 27 where the water

depth was approximately 25 ft, exceeding the ideal minimum depth— 2.5 times the propeller diameter— under the prop by approximately 6.5 ft, and there was negligible current. For each test condition, the engines were run for 30 minutes after they stabilized. Each test was repeated four times. Analysis determined that the repeatability of engine speed and power for an event series, and resulting confidence intervals, varied because of miscellaneous debris in the water below the vessel which could wrap around the propeller during the testing. This variable, for example, caused significant nonuniformity in loads. Therefore, the data was divided into segments of repeatable loads observed during the bollard testing.

Composite-events test design for the *Pathfinder* specified a series of in-use engine power conditions while operating in a canal with very little current. However, it was found that the repeatability of engine speed and power for an event series, and resulting confidence intervals, varied because of unknowns attached to the *Pathfinder* propeller during the testing. As a result, only segments of the composite-events data, representing stable operation of the engine and consistent loading at each load level while transiting up and down the canal, were utilized for data analysis. While in transit, propeller cavitations were not a problem, and the analyzed data segments were for engine loads of 25% (1132 rpm), 50% (1425 rpm), 75% (1630 rpm), and 100% (1800 rpm).

Performance of the additives was evaluated by determining the differences between *baseline* (no additive) and *candidate* (with additive) brake specific fuel consumption (BSFC) for each test condition. BSFC is the rate of fuel consumption divided by the power produced. For the engine-generator sets, it is given as pounds of fuel per kilowatt per hour (lb/kWh), and for the *Pathfinder* as pounds of fuel per brake horse power per hour (lb/bhp-hr).

Coriolis meters installed in the diesel engines' fuel supply and return lines measured the fuel mass flow rate. Fuel consumption is the difference between the supply and return fuel rates. Technicians secured the meters to engine room supports with mounting assemblies to isolate them from engine vibration. The return meter was installed vertically with the flow moving from the bottom to top to quickly remove any bubbles in the fuel that may have formed due to excess injector pump temperatures. A ball valve was installed at the discharge of the flow meter to minimize air bubbles in the coriolis tube. Once the meters were installed, test personnel

ensured that the engines ran properly. Fuel from the return meter was briefly diverted into a transparent hose or a bucket to verify that the flow was not aerated. Prior to commencing testing, operators stopped all engines and technicians completed zero flow checks to the fuel-filled meters per the manufacturer's specifications.

For the Lock and Dam No. 4 and the *Taggatz* engine-generator sets, an ION 7600 power meter was used to gather electrical power and energy data from the generator sets. Voltage lines for the power meter were connected directly to the lines going to the load bank, and electrical current data was collected from a set of 400:1 current transformers that were also installed on the load-bank lines. The ION meter calculates electrical power, power quality, and energy data from these connections. A pulse signal was sent from the ION meter to the data logger for every 300 Wh of electrical energy generated to verify consumption by the load bank.

For the *Pathfinder*, a Binsfield TorqueTrack 10K (TT10K) telemetry system with strain gauges bonded to each propeller shaft was used to measure engine-specific power. Shaft revolutions were measured by a Binsfield rpm module with magnetic sensors. The torque sensor consists of a strain gauge permanently bonded to the drive shaft and a battery-powered transmitter. Test personnel attached the transmitter and its 9-volt battery back to the shaft during the monitoring periods. Shaft diameter and material were inputs to the power calculations.

The shaft revolutions sensor was mounted on an adjustable magnetic base that was attached to a steel plate below the propeller shaft. Two permanent magnets were attached to the shaft. An acquisition module converted the analog and pulse signals to RS-232 to allow communication with the data logging computer. One-second (1 Hz) in-use fuel consumption, torque, shaft rpm, and power production data were logged during the tests.

## 4 Fuel Additives

At Lock and Dam No. 4, an alcohol injection system designed by Austin Renewable Fuels was installed on the generator engine. The injection system claims a boost in diesel performance, improvements in fuel economy, and a reduction in emissions. It is designed to deliver a 50/50 ethanol and water mixture to the engine's intake system after the turbo. Its major components consist of a controlled pump and three injection nozzles that were installed in the engines' intake manifolds. The typical maximum injection rate for the generator engine at the Lock and Dam No.4 was approximately 6 gallons per hour. The ethanol alcohol properties are listed in Table 1.

Table 1. Ethanol properties.

Appearance	Clear liquid
Odor	Sweetish, pleasant
Specific gravity at	0.875 @ 60 °F
Density	0.801 g/cm <sup>3</sup> @ 60 °F
Flashpoint (t.c.c.)	55.4 °F
Boiling range	78.1 °F
Water content	<0.5%

A diesel fuel catalyst (DFC) produced by Envirofuels was tested on the *Tagatz*. The Envirofuels DFC claims to provide improvements in fuel economy by creating a catalytic reaction that optimizes the heat release rate of the fuel which leads to an increase in power, reduced emissions, and a decrease in fuel consumption. It is also stated to create a nonaccumulative surface conversion which forms through chemisorptions of the inorganic polymer complex into the surface of ferrous and nonferrous metals. The polymer complex is said to passivate the surface, improving reflectivity and reducing oxygen reactivity, which results in more complete combustion.

The DFC was blended with the vessel's number one diesel and required a minimum degreening period<sup>1</sup> of 300 hr per engine before candidate testing would be able to measure its effects. After the baseline tests were

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<sup>1</sup> A degreening period is the amount of time required to obtain a stable catalyst prior to assessing its performance characteristics.

completed, the onboard fuel tanks were filled with diesel fuel, and the DFC was added to produce the recommended ratio of 625:1 by volume for the initial passivation and degreening period. Before the additive testing began, the recommended ratio of 1250:1 was run in the engines for more than a month (only a 7-day period was required by the additive manufacturer). Table 2 lists the fuel and additive amounts during the degreening period. Table 3 lists the DFC properties.

**Table 2. Additives (DFC) and fuel amounts during the degeening period.**

Date	Fuel (gallons)	DFC (gallons)	Ratio
8/3/10	3458	5.5	629:1
8/9/10	2450	3.75	653:1
8/23/10	2302	3.75	614:1
8/31/10	2520	4.0	630:1
9/8/10	2650	2.0	1325:1
9/18/10	2618	2.0	1309:1
9/25/10	2260	1.8	1256:1

**Table 3. DFC properties.**

Appearance	Translucent reddish liquid
Specific gravity at	0.85 @ 60 °F
Density	N/A @ 60 °F
Flashpoint (COC)	>212 °F
Viscosity @ 212 °F	4.4-5.7 cSt
API Gravity or Density	N/A

Two additive technologies were selected for *Pathfinder* testing. The DurAlt fuel additive claims to boost diesel performance by decreasing the engine's requirement for cetane level, resulting in improved fuel economy of 10% to 20%. It also claims to lubricate and clean injectors and reduce emissions. The recommended dosage for the DurAlt fuel additive (as provided by the manufacturer) is 1 oz per 23 gallons of diesel fuel. The physical properties of DurAlt fuel saver are listed in Table 4.

The Lucas Fuel Treatment additive produced by Lucas Oil Products, Inc. claims to provide improvements in fuel economy and has additives with high detergents for cleaning the fuel system and combustion chamber. The recommended dosage for the Lucas additive (as provided by the manufacturer) is 32 oz (1 quart)/946 ml per 100 gallons of fuel. The physical properties of the Lucas fuel additive are listed in Table 5.

**Table 4. DurAlt Fuel Conditioner properties.**

Appearance	Translucent, amber liquid
Odor	Sweetish, distinctive
Specific Gravity at	0.875 @ 60 °F
Density	7.33 @ 60 °F
Flashpoint (t.c.c)	150 °F
Boiling Range	220–600 °F
Water Content	< 0.5%

**Table 5. Lucas Fuel Treatment additive properties.**

Appearance	Translucent, yellow liquid
Specific Gravity at	0.8961 @ 60 °F
Density	7.462 @ 60 °F
Flashpoint (COC)	460 °F
Viscosity @ 212 °F	13
API Gravity	26.4

Approximately 1,000 gallons of fuel blended with each additive was used in the engines before testing began. Table 6 lists the fuel and additive amounts that were used in the break-in periods and the manufacturers' recommended ratios.

**Table 6. Additives and fuel for break-in periods and testing.**

Date	Lucas Fuel Additive		Recommended Additive Amount (quarts)	Date	DurAlt Fuel Additive		Recommended Additive Amount (oz)
	Fuel (gallons)	Additive (quarts)			Fuel (gallons)	Additive (oz)	
09/03/09	300	3	3.0	9/12/09	546	23	23.296
09/04/09	510	5	5.1	9/14/09	142	6.85	6.05
09/04/09	213	2	2.13	9/15/09	342	16	14.592
09/05/09	296	3	2.96	9/16/09	312	13	13.312
Total	1319	13	13.19	9/17/09	181	8.5	7.72
				Total	1523	67.35	64.97

## 5 Test Results

### Lock and Dam No. 4

The results for Lock and Dam No. 4 are given in Tables 7, 8 and 9, and shown in Figures 3, 4, and 5.

During testing, the return fuel flows could not accurately be measured at loads below 50% with the alcohol injection system operating. During the 25% load tests, diesel fuel use was diminished to the point that it was boiling in the return system, and the data were highly inconsistent, with typical candidate fuel-use standard deviations of 15% of the base fuel consumption reading. Consequently, the results of the 25% load tests were excluded. The percentage changes in fuel consumption for the load bank tests are listed for both the baseline tests as compared to the additive tests, and for the changes between each of the baseline fuel test results.

**Table 7. Lock and Dam No. 4 load-bank test results, 50% load.**

	50% Load Test		
	Baseline 1	Candidate	Baseline 2
AVERAGE (lb/kWh)	NA	0.406	0.565
STDEV (lb/kWh)	NA	0.000	0.017
Candidate Difference vs. Baseline (lb/kWh)	NA	-	-0.159
Candidate % Difference vs. Baseline	NA	-	-28.14%
Statistically Significant Difference?	NA	-	YES
95% Confidence Interval (lb/kWh)	NA	-	0.005

Note: Baseline 1 fuel return had bubbles in the return fuel line, invalidating the data.

**Table 8. Lock and Dam No. 4 load-bank test results, 75% load.**

	75% Load Test		
	Baseline 1	Candidate	Baseline 2
AVERAGE Diesel BSFC (lb/kWh)	0.530	0.464	0.535
STDEV Diesel BSFC (lb/kWh)	0.001	0.001	0.002
Candidate Difference vs Baseline (lb/kWh)	-0.066	-	-0.072
Candidate % Difference vs. Baseline	-12.44%	-	-13.36%
Statistically Significant Difference?	YES	-	YES
95% Confidence Interval (lb/kWh)	0.003	-	0.005

Table 9. Lock and Dam No. 4 load-bank test results, 90% load.

	90% Load Test		
	Baseline 1	Candidate	Baseline 2
AVERAGE Diesel BSFC (lb/kWh)	0.504	0.443	0.509
STDEV Diesel BSFC (lb/kWh)	0.003	0.001	0.006
Candidate Difference vs Baseline (lb/kWh)	-0.060	-	-0.066
Candidate % Difference vs. Baseline	-11.99%	-	-12.96%
Statistically Significant Difference?	YES	-	YES
95% Confidence Interval (lb/kWh)	0.005	-	0.012

Figure 3. Lock and Dam No. 4 load-bank test results, 50% load.

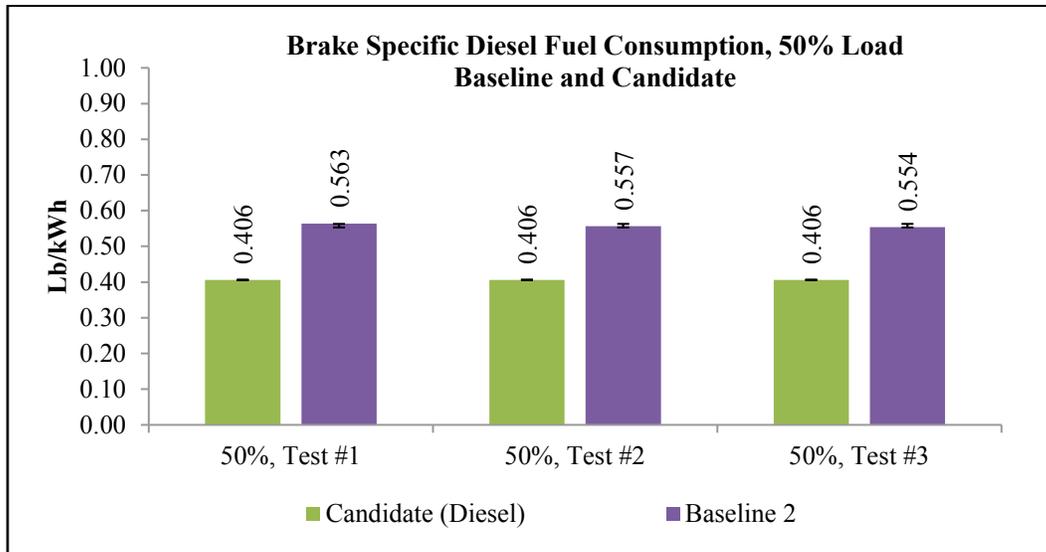


Figure 4. Lock and Dam No. 4 load-bank test results, 75% load.

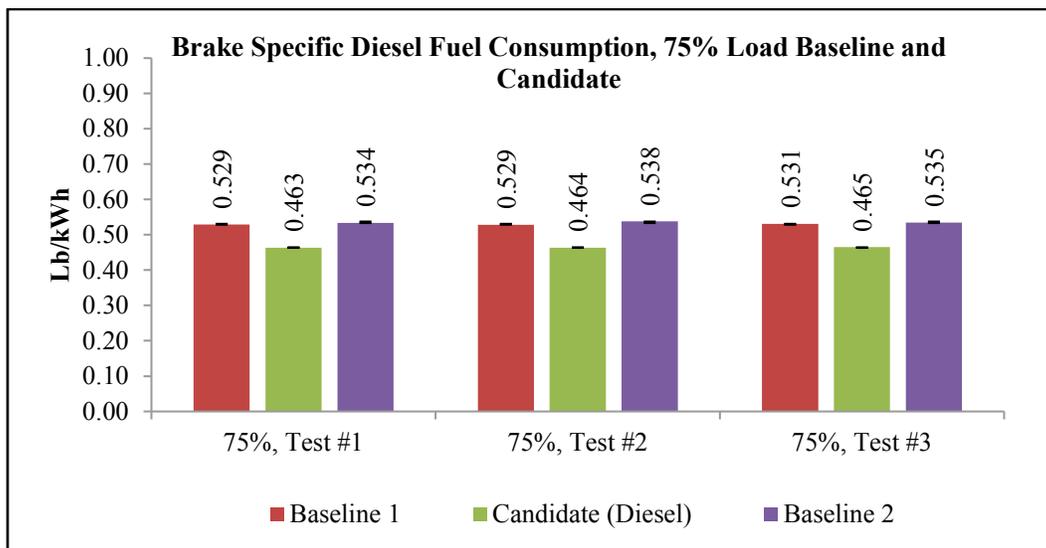
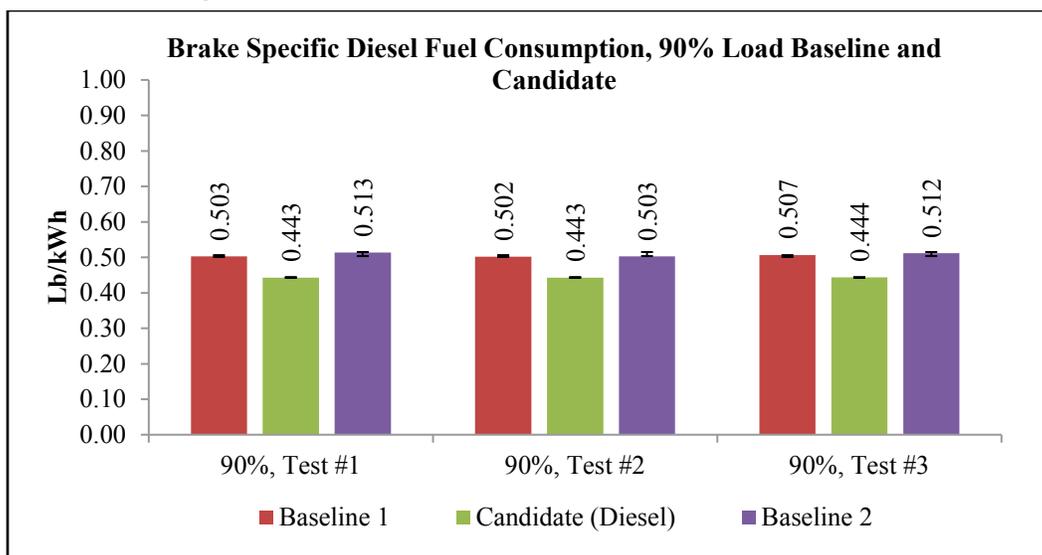


Figure 5. Lock and Dam No. 4 load-bank test results, 90% load.



The negative numbers indicate a decrease in fuel usage over the baseline fuel. There are statistically significant differences in diesel fuel consumption when using the candidate alcohol injection system compared to baseline usage at the 50% load (a reduction of approximately 30%), and at the 75% and 90% generator loads (a reduction of approximately 12% for both). The higher reduction in diesel fuel usage at the 50% load is likely because the injection system did not dose alcohol according to any engine load parameters but rather maintained the same injection pressure at all engine loads. Because of this constant injection pressure, alcohol flow was slightly reduced at higher load settings due to increased back pressure in the intake manifold, where the injection took place. As a result of these effects, proportionally more diesel was displaced at 50% load than at the 75% and 90% loads (as reflected by the aforementioned fuel consumption reduction values at different loads).

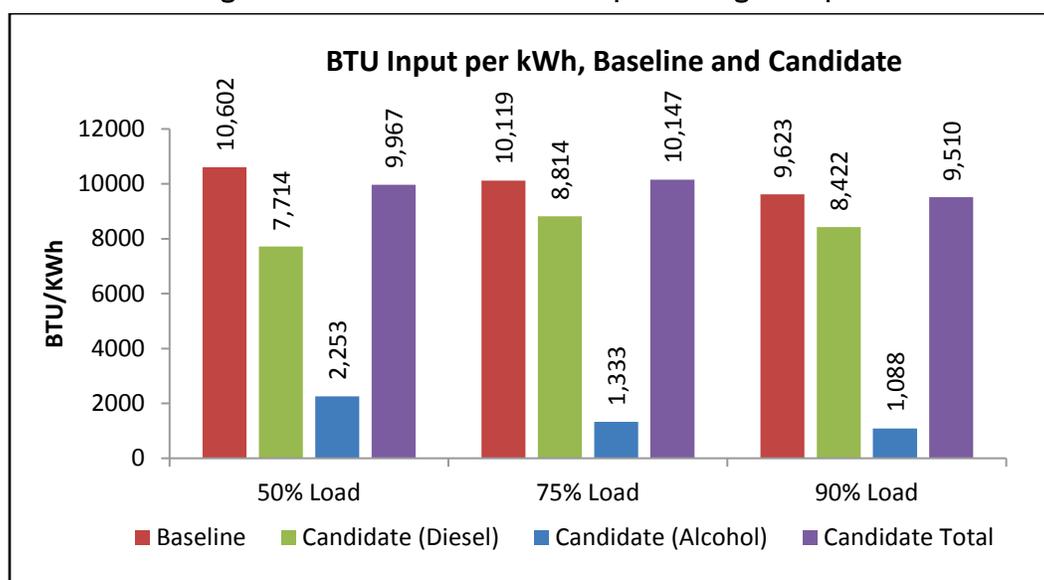
To determine if using the alcohol injection system had any effect on carbon dioxide (CO<sub>2</sub>) emissions, CO<sub>2</sub> emissions per megawatt-hour were estimated. Emissions factors were estimated using the Department of Energy, Specific Energy, Energy Density, and CO<sub>2</sub> data for diesel and alcohol. The estimated values are listed in Table 10. Negative numbers indicate a decrease in CO<sub>2</sub> production while positive numbers represent an increase. The calculations show decreases in CO<sub>2</sub> production at the 50% and 90% loads and little or no difference at the 75% load.

Table 10. Lock and Dam No. 4 estimated CO<sub>2</sub> emission differences.

Estimated CO <sub>2</sub> Emissions	50% Load	75% Load	90% Load
Baseline Diesel Emissions (lb-CO <sub>2</sub> /MWh)	1822	1739	1653
Candidate Diesel Emissions (lb-CO <sub>2</sub> /MWh)	1325	1514	1447
Candidate Alcohol Emissions (lb-CO <sub>2</sub> /MWh)	385	228	186
Candidate Total Emissions (lb-CO <sub>2</sub> /MWh)	1711	1742	1633
Candidate Difference, %	-6.08%	0.21%	-1.22%

The lock and Dam No. 4 energy output in kWh was compared with the engine's energy input to determine if there was an improvement in engine efficiency when using the alcohol injection system. Figure 6 shows the average BTU input per fuel type and subsequent kWh output for

Figure 6. Lock and Dam No. 4 BTU input and engine output.



each test load during baseline and candidate testing. In the figure, the first bar represents the BTU content of the fuel used during the baseline conditions. The second and third bar in the graph represent the diesel and alcohol fuel BTU content used during the candidate conditions, and the final bar represents the total BTU of fuel used during the candidate conditions. The calculations were based on BTU contents found on the GREET Transportation Fuel Cycle Analyses Model (Argone National Laboratory 2008). The BTU content of a gallon of ethanol and diesel used to calculate the results is 76,300 and 128,450, respectively. The engine is more efficient with the candidate system at the 50% and 90% loads by 5.98% and 1.17%, respectively, and less efficient at the 75% load by 0.27%.

## Taggatz

The test loads for the *Taggatz* are listed for port engine-generator set in Table 11 and for the starboard engine-generator set in Table 12. The percentages are the percent differences from the target test design loads. For the port engine, the data for the 25% load were eliminated from the analysis because of the difference between baseline and candidate engine loading. The discrepancy was the result of a miscommunication between the test personnel and the load-bank operator.

Table 11. *Taggatz* port test loads.

Test Load	Port Baseline Load Average	Baseline Load % from Target	Target Load	Port Candidate Load Average	Candidate Load % from Target
% of Max (330kW)	kW	%	kW	kW	%
25	68.37	-17.12%	82.5	83.90	1.69%
50	161.16	-2.33%	165	165.55	0.33%
75	245.02	-1.00%	247.5	246.50	-0.40%
85	270.89	-3.43%	280.5	274.81	-2.03%

Table 12. *Taggatz* starboard test loads.

Test Load	Starboard Baseline Load Average	Starboard Baseline Load % from Target	Target Load	Starboard Candidate Load % from Target	Starboard Candidate Load Average
% of Max (330kW)	kW	%	kW	%	kW
25	81.24	-1.53%	82.5	1.98%	84.14
50	164.86	-0.08%	165	1.13%	166.86
75	244.75	-1.11%	247.5	-0.01%	247.48
85	280.29	-0.07%	280.5	1.34%	284.27

The port test results are listed in Tables 13, 14, and 15 and shown in Figures 7, 8, and 9, and the starboard test results are shown in Tables 16, 17, 18, and 19 and Figures 10, 11, 12, and 13. Positive numbers indicate an increase in fuel usage over baseline fuel usage, and negative numbers indicate a decrease in fuel usage over baseline fuel usage.

Table 13. *Taggatz* port test results, 50% load.

	50% Load Test	
	Baseline 1	Candidate
AVERAGE (lb/kWh)	0.532	0.512
STDEV (lb/kWh)	0.002	0.001
Candidate Difference vs. Baseline (lb/kWh)	0.020	-
Candidate % Difference vs. Baseline	-3.73%	-
Statistically Significant Difference	YES	-
95% Confidence Interval (lb/kWh)	0.003	-

Table 14. *Taggatz* port test results, 75% load.

	75% Load Test	
	Baseline 1	Candidate
AVERAGE Diesel BSFC (lb/kWh)	0.502	0.495
STDEV Diesel BSFC (lb/kWh)	0.001	0.000
Candidate Difference vs Baseline (lb/kWh)	0.006	-
Candidate % Difference vs. Baseline	-1.28%	-
Statistically Significant Difference?	YES	-
95% Confidence Interval (lb/kWh)	0.007	-

Table 15. *Taggatz* port test results, 85% load.

	75% Load Test	
	Baseline 1	Candidate
AVERAGE Diesel BSFC (lb/kWh)	0.506	0.458
STDEV Diesel BSFC (lb/kWh)	0.010	0.003
Candidate Difference vs Baseline (lb/kWh)	0.048	-
Candidate % Difference vs. Baseline	-9.51%	-
Statistically Significant Difference?	YES	-
95% Confidence Interval (lb/kWh)	0.021	-

Figure 7. *Taggatz* port generator baseline and candidate BSFC, 50% load.

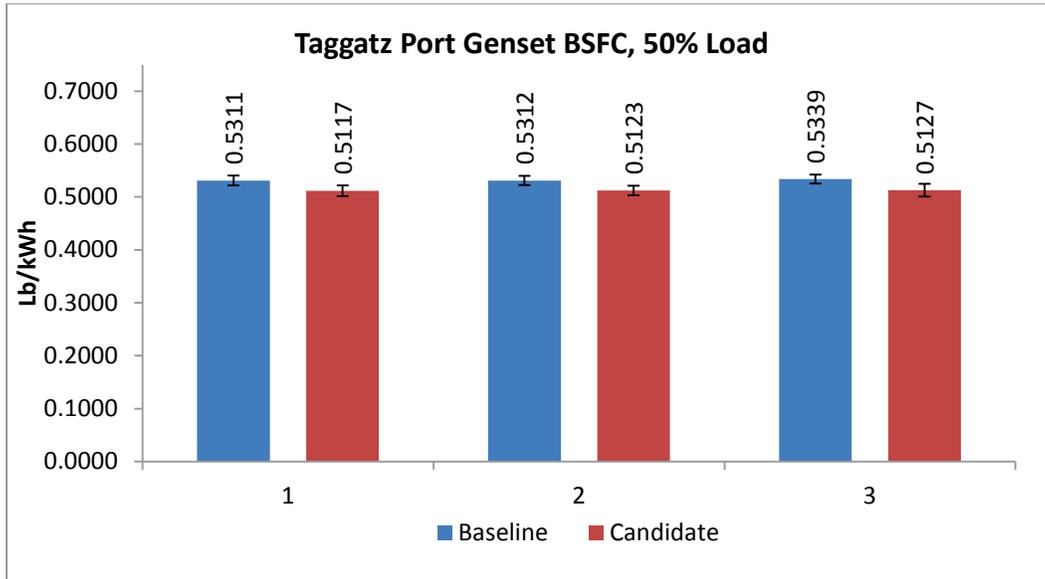


Figure 8. *Taggatz* port generator baseline and candidate BSFC, 75% load.

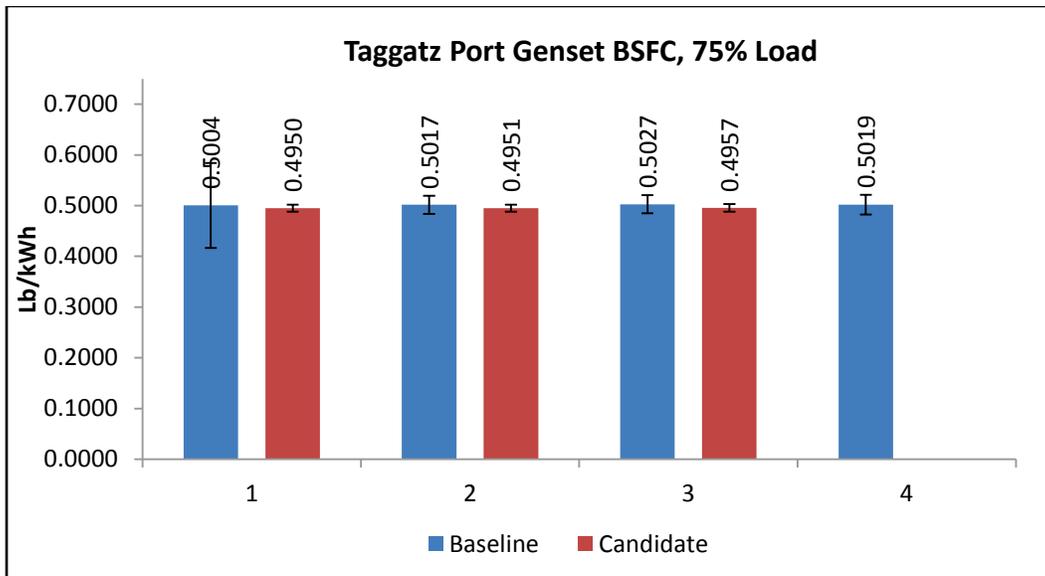


Figure 9. *Taggatz* port generator baseline and candidate BSFC, 85% load.

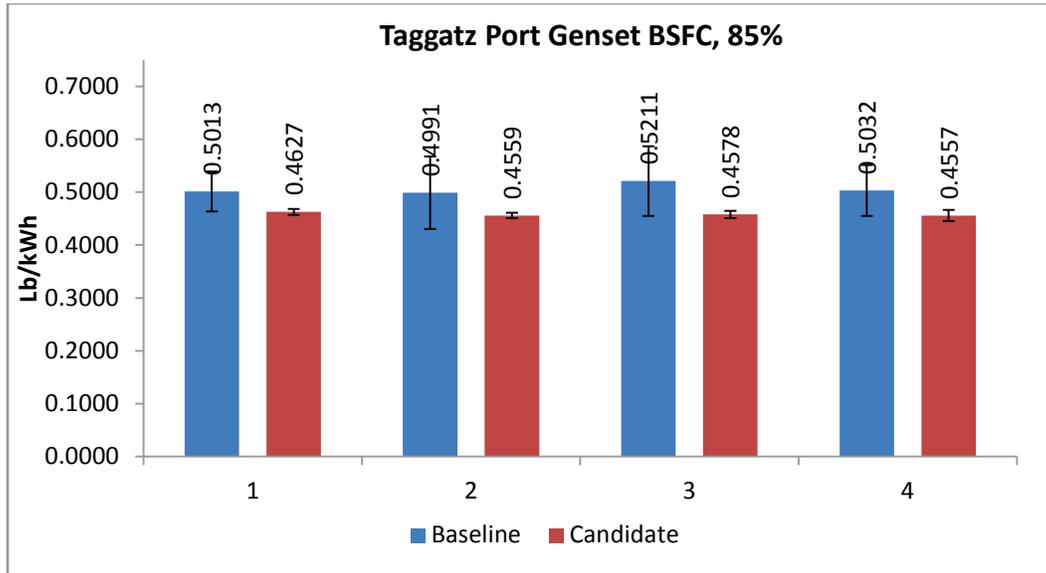


Table 16. *Taggatz* starboard test results, 25% load.

	25% Load Test	
	Baseline 1	Candidate
AVERAGE (lb/kWh)	0.615	0.636
STDEV (lb/kWh)	0.006	0.006
Candidate Difference vs Baseline (lb/kWh)	0.020	-
Candidate % Difference vs. Baseline	3.32%	-
Statistically Significant Difference?	YES	-
95% Confidence Interval (lb/kWh)	0.016	-

Table 17. *Taggatz* starboard test results, 50% load.

	50% Load Test	
	Baseline 1	Candidate
AVERAGE (lb/kWh)	0.515	0.520
STDEV (lb/kWh)	0.003	0.003
Candidate Difference vs Baseline (lb/kWh)	0.007	-
Candidate % Difference vs. Baseline	1.02%	-
Statistically Significant Difference?	No	-
95% Confidence Interval (lb/kWh)	0.007	-

Table 18. *Taggatz* starboard test results, 75% load.

	75% Load Test	
	Baseline 1	Candidate
AVERAGE Diesel BSFC (lb/kWh)	0.494	0.501
STDEV Diesel BSFC (lb/kWh)	0.000	0.007
Candidate Difference vs Baseline (lb/kWh)	0.007	-
Candidate % Difference vs. Baseline	1.33%	-
Statistically Significant Difference?	No	-
95% Confidence Interval (lb/kWh)	0.015	-

Table 19. *Taggatz* starboard test results, 85% load.

	85% Load Test	
	Baseline 1	Candidate
AVERAGE Diesel BSFC (lb/kWh)	0.489	0.496
STDEV Diesel BSFC (lb/kWh)	0.001	0.002
Candidate Difference vs Baseline (lb/kWh)	0.007	-
Candidate % Difference vs. Baseline	1.51%	-
Statistically Significant Difference?	YES	-
95% Confidence Interval (lb/kWh)	0.004	-

Figure 10. *Taggatz* starboard generator and candidate baseline BSFC, 25% load.

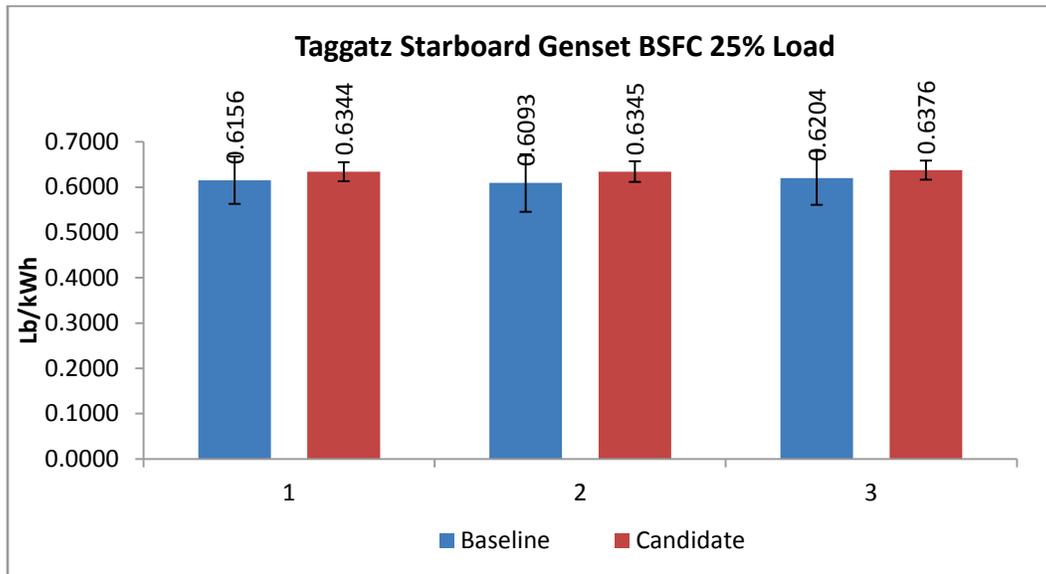


Figure 11. *Taggatz* starboard generator baseline BSFC, 50% load.

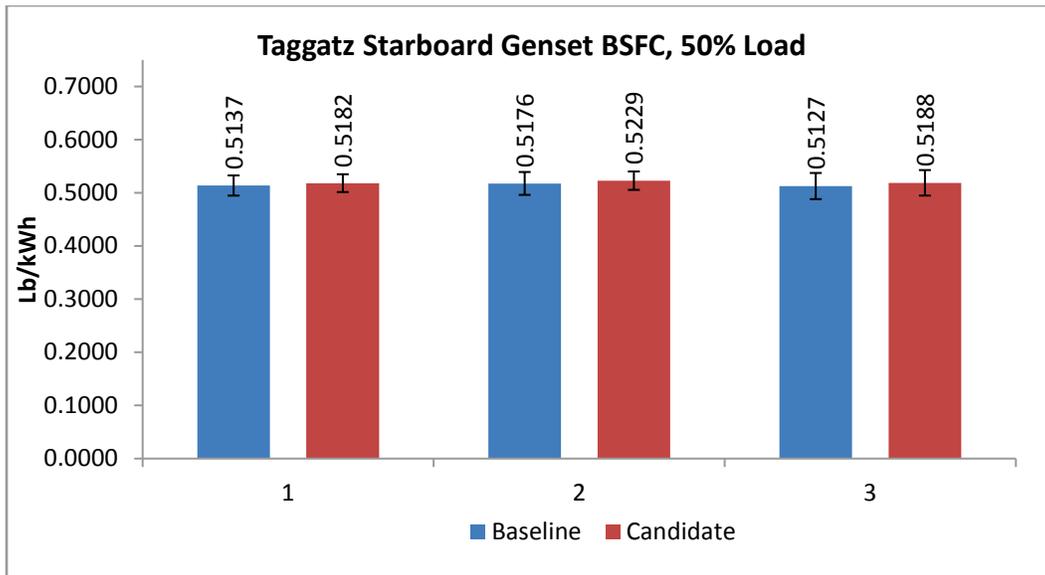


Figure 12. *Taggatz* starboard generator baseline BSFC, 75% load.

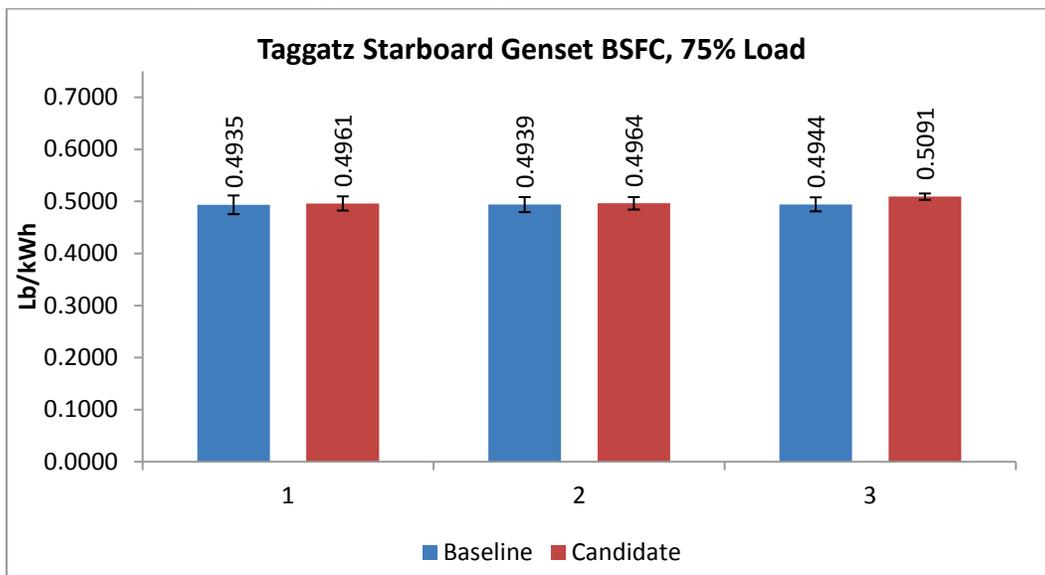
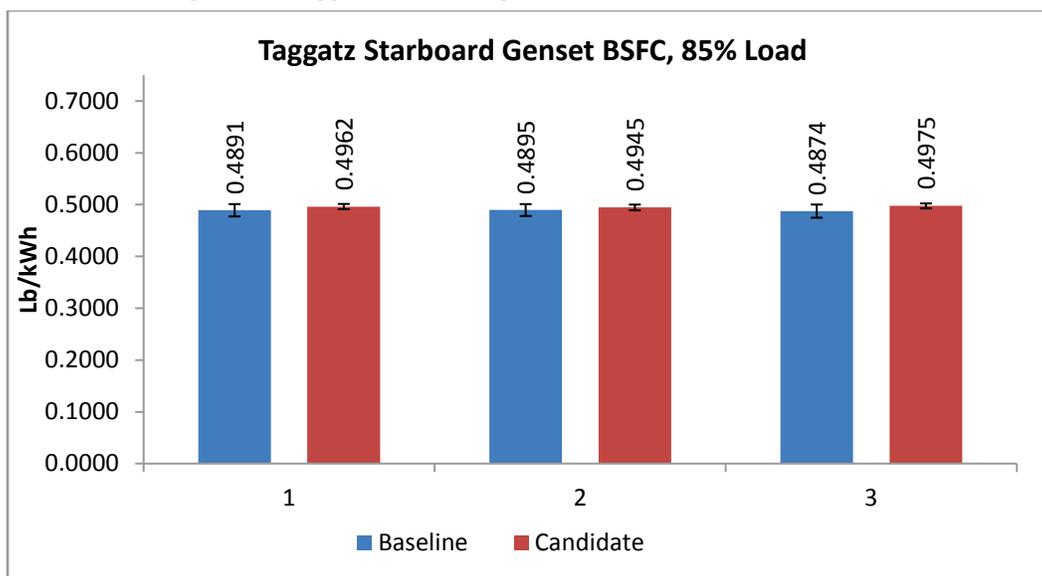


Figure 13. *Taggatz* starboard generator baseline BSFC, 85% load.

All of the port-side results were shown to be statistically significant. However, for the starboard side, only the 25% and 85% load tests, which showed a slight increase in fuel consumption during the additive testing, are statistically significant. The candidate additive shows fuel use decreases of 3.73%, 1.28%, and 9.51% for port engine loads of 50%, 75%, and 85%, and increases of 3.32%, 1.02%, 1.13%, and 1.51% for starboard engine loads of 25%, 50%, 75%, and 85%.

The port engine's governor actuator required replacement after the baseline testing had been performed and also required slight adjustments during the degreening period. The actuator helps the engine maintain a constant speed of 1800 rpm, thereby allowing the generator to produce a steady amount of energy. After the degreening period had begun, engine speed variation was noticed, and the necessary parts were replaced.

The starboard engine's fuel and load data were consistent, and data collection and analyses were conducted without problems. Since the starboard engine showed a statistically significant increase in fuel consumption during candidate testing at the 85% load, the 9.51% decrease in fuel consumption at the 85% load for the port engine is discounted. Overall, the conclusions are that the starboard engine results are the reliable ones and that the additive did not reduce fuel consumption. The problem with the port engine results may be the result of an improperly tuned governor actuator.

## Pathfinder

The bollard-push test results for the *Pathfinder* are presented in Figure 14 and Table 20. As illustrated in Figure 14, at each load condition, little change is observed in the fuel consumption from baseline to candidate. In Table 20, the percent change in fuel consumption for the bollard push tests are listed for the baselines as compared to the additive and the changes between each of the baseline fuel test results. Comparisons are made between the fuel consumption during baseline 1 (BL1) or baseline 2 (BL2) and candidate 1 (C1 – Lucas) and separately between baseline 2 (BL2) or baseline 3 (BL3) and candidate 2 (C2 – DurAlt). Positive numbers indicate a slight increase in fuel usage over the baseline fuel usage. The negative numbers indicate a decrease in fuel usage compared to that of the baseline fuel. No statistics could be performed on these results because these tests were performed as a single test for a long duration of time.

Figure 14. *Pathfinder* bollard push test results at different loads.

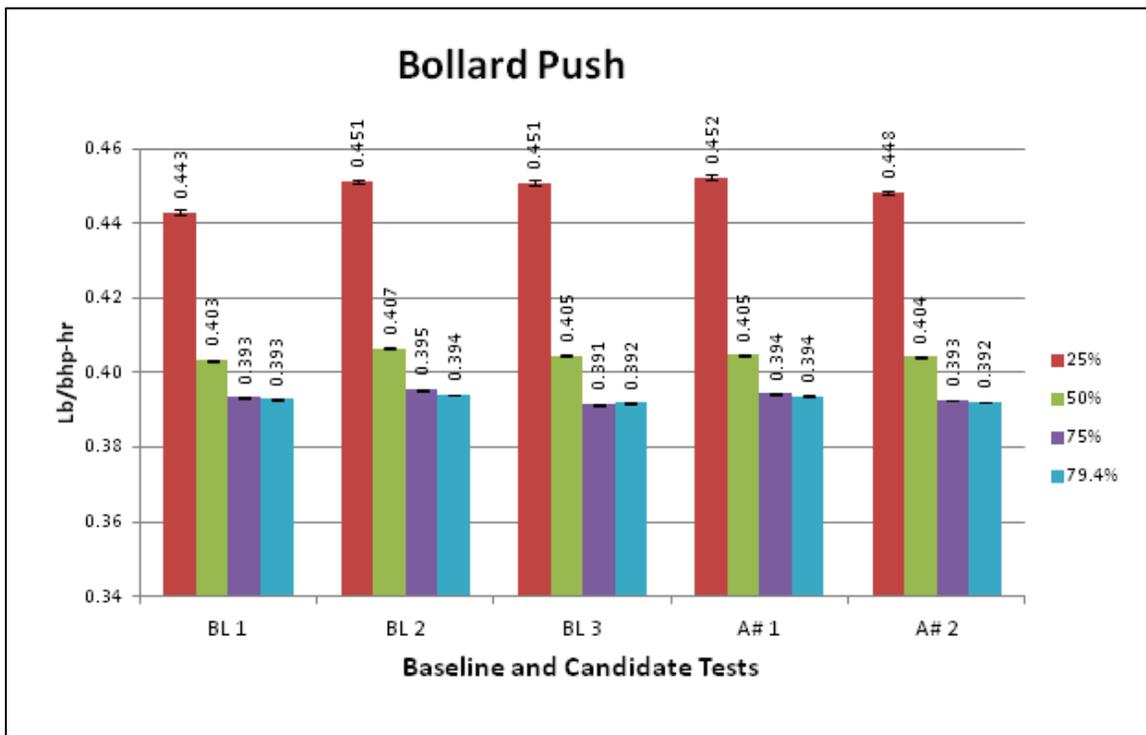


Table 20. *Pathfinder* bollard-push baseline and candidate fuel consumption changes.

Load %	Percent Change BL1 to C1	Percent Change BL2 to C1	Percent Change BL1 to BL2	Percent Change BL2 to C2	Percent Change BL2 to BL3	Percent Change BL3 to C2
25	2.0896	0.2565	1.8378	-0.6680	-0.0699	-0.5977
50	0.3325	-0.4675	0.7963	-0.5741	-0.4950	-0.0787
75	0.2268	-0.2953	0.5206	-0.7408	-1.0103	0.2668
100	0.2309	-0.0869	0.3176	-0.5050	-0.5651	0.0598

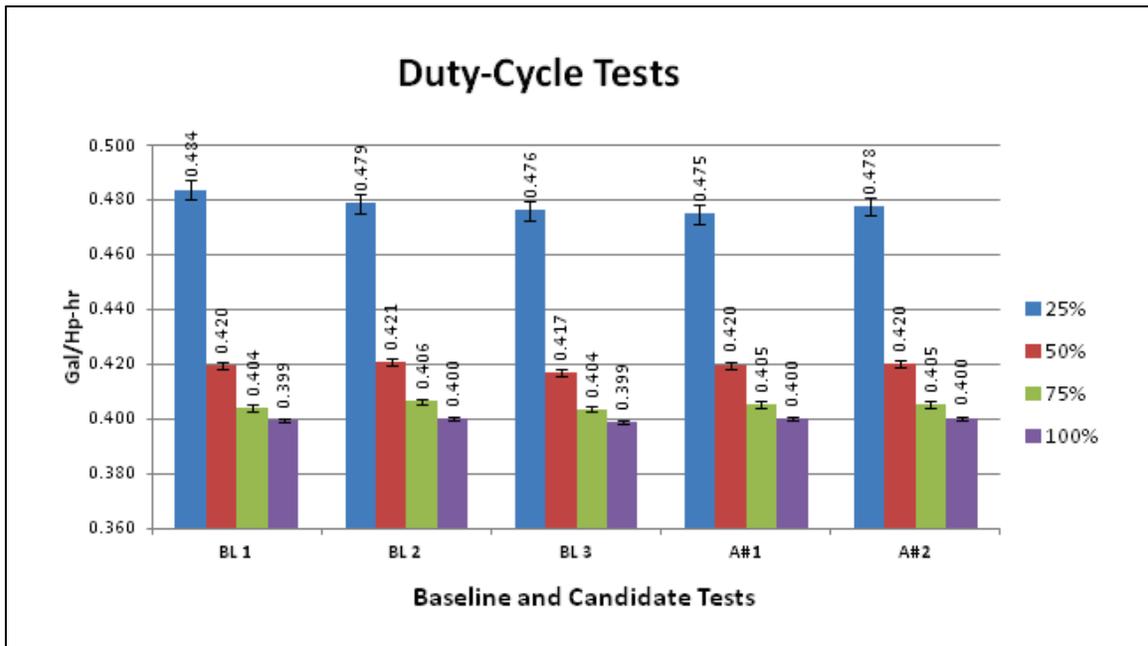
The duty-cycle test results for the *Pathfinder* are presented in Table 21 and Figure 15. As presented in Table 21 and Figure 15, the impacts of the additives on fuel consumption appear to be minimal for the operating regimes under which the tests were completed. Statistical significance was calculated using a two-tailed t-test evaluation. Only one difference was found to be statistically different, the BL1 to C1 difference of -1.8924 at 25% load.

It should be noted that the in-use tests were completed while traveling downstream and then while traveling upstream. Because of the impacts of the current on engine power delivered to the shaft, there were slight differences between the fuel consumption when operating in each direction. The data presented in Table 21 and Figure 15 aggregate all test runs (two each of up and downstream) into a single mean fuel consumption rate and associated standard deviation. However, as a worst case, the statistical significance of any changes between additive-dosed and baseline fuels for upstream-only and downstream-only data was evaluated. In each case, the slight decrease of the standard deviation due to the elimination of the downstream data (and vice versa) did not impact the statistical significance of any observed changes.

Table 21. *Pathfinder* duty-cycle baseline and candidate fuel consumption changes.

Load %	Percent Change BL1 to C1	Percent Change BL2 to C1	Percent Change BL1 to BL2	Percent Change BL2 to C2	Percent Change BL2 to BL3	Percent Change BL3 to C2
25	-1.8924	-0.8321	-1.0515	-0.2160	-1.0053	0.3126
50	0.0076	-0.2339	0.2409	-0.1952	-1.2169	0.6669
75	0.2742	-0.2659	1.7230	-0.2746	-1.1072	0.4175
100	0.2032	0.0005	0.2027	0.0721	-0.4695	0.3594

Figure 15. Duty-cycle test results at different loads.



## 6 Summary and Conclusions

Four fuel additives were tested to evaluate their potential for reducing diesel fuel consumption and cost:

- An ethanol injection system
- Envirofuels Diesel Fuel Catalyst
- DurAlt Fuel Conditioner
- Lucas Fuel Treatment.

Fuel usage was measured while using the additives with diesel fuel (candidate tests) and compared to fuel usage under the same conditions while using only standard diesel fuel (baseline tests). The tests were conducted on diesel generator sets at the USACE Lock and Dam 4 Service Base (ethanol injection) in Alma, WI, the quarters boat *Taggatz* (Envirofuels Diesel Fuel Catalyst) located near Wabasha, MN, and the towboat *Pathfinder* (DurAlt Fuel Conditioner and Lucas Fuel Treatment) located in Saint Louis, MO. The evaluations were conducted in the field under actual, in-use operating conditions. The results are applicable to the host engines and operating conditions, but similar results can be expected for similar engine families, year of manufacture, and operating regimes. While the Envirofuels, DurAlt, and Lucas additives showed limited fuel reduction in select operation conditions, only the ethanol injection system consistently showed potential to reduce diesel fuel consumption.

During the candidate test runs, diesel fuel consumption using the ethanol injection system was reduced by an average of 30% at the 50% loads, and 12% at the 75% and 90% loads. The lower reduction in diesel fuel usage at the higher load levels is likely due to the injection system's static flow rate and sensitivity to turbocharger boost pressures. Instead of maintaining the same alcohol mass flow at all engine loads, this caused a decrease in the alcohol dosing at the higher power levels. As a result, proportionally more diesel was displaced at 50% than at the 75% and 90% loads. The engine was found to be more efficient with the candidate system at the 50% and 90% loads by 5.98% and 1.17%, respectively, and less efficient at the 75% load by 0.27%.

While using the alcohol injection system, it was found that CO<sub>2</sub> emissions were reduced by 6.08% for the 50% load, 0.21% at the 75% load and 1.22% at the 90% load. It is possible that if the alcohol injection amount had remained constant throughout the different loading conditions or had a more variable flow rate that scales proportionally with diesel use, the higher load results would have more resembled the data collected at the 50% load condition.

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# REPORT DOCUMENTATION PAGE

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