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THE DEVELOPMENT OF NONLINEAR ANALYSIS METHODS FOR BONDED JOINTS IN ADVANCED FILAMENTARY COMPOSITE STRUCTURES

> G. C. Grimes L. F. Greimann T. Wah G. E. Commerford W. R. Blackstone G. K. Wolfe



Southwest Research Institute

**TECHNICAL REPORT AFFDL-TR-72-97** 

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Air Force Flight Dynamics Laboratory Air Force Systems Command Wright-Patterson Air Force Base, Ohio

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AFFDL-TR-72-97

# THE DEVELOPMENT OF NONLINEAR ANALYSIS METHODS FOR BONDED JOINTS IN ADVANCED FILAMENTARY COMPOSITE STRUCTURES

G. C. Grimes L. F. Greimann T. Wah G. E. Commerford W. R. Blackstone G. K. Wolfe

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## FOREWORD

The research reported herein was performed by Southwest Research Institute, 8500 Culebra Road, Box 28510, Sun Antonio, Texac 78284 in the Department of Structural Research under Air Force Contract No. F33615-69-C-1041. The contract was initiated under Project No. 436403 by the Air Force Flight Dynamics Laboratory, Air Force Systems Command, Wright-Patterson Air Force Base, Ohio 45433. Dr. Edvins Demuts, Air Force Project Engineer administered the program.

This report covers the research work conducted at SwRI between 14 April 1969 and 30 October 1971 under the management of Mr. Glenn C. Grimes, Project Leader in the Department of Structural Research headed by Dr. Robert C. DeHart, Director and Mr. L. U. Rastrelli, Assistant Director. Principal Investigators in this effort were Dr. L. F. Greimann, Dr. T. Wah,\* Mr. G. E. Commerford, Mr. W. R. Blackstone, and Mr. G. K. Wolfe.

The authors gratefully acknowledge the fine laboratory work of Senior Technician Mr. William Keith and Technician Mr. Albert Reichert under the direction of Mr. B. C. hoy, Laboratory Manager.

This technical report has been reviewed and is approved.

Philip A Garmley

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\*Associate Professor, School of Engineering, Texas A & I University, Kingsville, Texas.

#### ABSTRACT

Development of analysis methods for orthotropic adherend bonded lap joints which account for material nonlinearities at room temperature was the primary objective of the research reported herein. The use of these methods in predicting mechanical behavior, ultimate loads, and failure modes was the goal. In order to accomplish this, new analytical procedures were developed and successfully checked with discrete element techniques for single, double, and step lap adhesively bonded attachment configurations. Experimental verification of these nonlinear analyses was accomplished by the fabrication and evaluation of a variety of simple joint specimens under static monotonically increasing load. Failure loads and modes were used as the primary substantiation characteristics but the mechanical behavior of a small number of these simple joint specimens was observed at intermediate loadings and found to compare favorably with the analytically predicted behavior. Larger, more complex bonded joints were designed, fabricated, and evaluated under static monotonically increasing loads at room temperature utilizing these methods. Utimate toad, failure mode, and detailed strain behavior at any intermediate load were accurately predicted with the new analyses, as substantiated by experimental observations. These techniques were put into a computerized design/analysis program for structural application use and the program was used to generate bonded joint design allowable curves.

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#### SECTION I

## INTRODUCTION

This research program had three objectives as defined in Exhibit "A" of the Statement of Work of Contract F33615-69-C-1641. The first one was to develop new static structural analysis methods for bonded joints which account for the adhesive and composite (or metal) adherend nonlinear behavior under stress. Prediction of static failure in all principal modes was to be the goal of the analytical techniques developed under this first objective. Objective number two was to develop a method of determining "effective" adhesive properties for use in the analytical methods since true properties data are seldom available and not generally obtained from simple tests. In the third objective, useful static design curves were to be developed based on the new analysis techniques. The accomplishment of these objectives was the primary goal of the research, reported herein.

The "how, what, and why" results obtained in accomplishing these objectives make up the contents of this report. Work and results in completing the requirements of the first objective are given in Section II—Theoretical Methods Development, III—Discrete Element Analysis (Development), and IV—Comparison of Theoretical and Discrete Element Results. In Section V—Experimental Design, literature data were surveyed, analyzed, and used in test program development on adherend materials and bonded joints. Manufacturing and quality control of the composite materials is covered in Section VI—Laminate Processing, whereas Section VII—Laminate and Titanium Adherend Test Results, reports the experimental characterization values obtained. Lap shear assembly manufacture and postbond fabrication are covered in Section VIII—Bonded Joint Test Results. With the completion of the work reported in the program are reported in Section IX—Bonded Joint Test Results. With the completion of the work reported in the previous Sections it was possible to satisfy the second objective as covered in Section XI—Results. Conclusions, and Recommendations, covers the meaning and impact of the research effort with suggestions for related, pertinent future study of bonded joint structures.

After the List of References, the Appendices, except Appendix A, present the detailed information utilized in the program and are included for the reader's perusal if more data are needed.

In summary, the three objectives outlined above have been satisfied and the method developed can be used in R. T. static nonlinear design/analysis of adhesive bonded single, double, and step lap joints. With the publication of this report, computer programs based on this analytical method are available through the Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio.

#### SECTION II

#### THEORETICAL METHODS DEVELOPMENT

#### 11.1. GENERAL

This section contains the mathematical development of nonlinear analysis methods\* for bonded single, double, and step lap joints subjected to static loads at R.T. These three joints are shown in Figure 1, along with the coordinate systems, dimensions, and applied loads which will be utilized in the following developments. The joints are assumed to be sufficiently wide in the z direction (perpendicular to the plane of the paper) such that the material under load is in a state of plane strain; i.e., the normal strain  $\epsilon_z$  and the shear strains  $\gamma_{xz}$  and  $\gamma_{yz}$  are assumed to be zero. In general, the adherends may be either orthotropic (laminates) or isotropic, and may have different thicknesses which are constant for each adherend. The adhesive is assumed to be isotropic and of a constant thickness which is much smaller than the adherend thickness. The adherends are assumed to be flat plates in bending; i.e., normal stresses through the thicknesses ( $\sigma_y$ ) are neglected. Interlaminat shear is neglected. Laminates are assumed to be



 symmetrical about their middle surface. Further assumptions involving material behavior and assumptions peculiar to each joint will be discussed as they occur.

#### IL2. SINGLE LAP JOINT

#### II.2.a. Equilibrium Equations

The differential equations of equilibrium governing the behavior of a segment of a bonded single lap joint can be developed from the free bodies in Figure 2. (The joint is assumed to have a unit width in the z direction.) Summing forces in the x and y directions and moments in the x-y plane for adherends 1 and 2 gives

$$\frac{dN_1}{dx} = 0 \qquad \qquad \frac{dN_2}{dx} + \tau = 0 \qquad (1a)$$

$$\frac{dV_1}{dx} = 0 \qquad \qquad \frac{dV_2}{dx} = 0 \qquad (1b)$$

$$\frac{\mathrm{d}M_1}{\mathrm{d}x} - V_1 + \tau \frac{t_1}{2} = 0 \quad \frac{\mathrm{d}M_2}{\mathrm{d}x} - V_2 - \tau \frac{t_2}{2} = 0 \tag{1c}$$

where  $\sigma$  and  $\tau$  are the normal stress and shear stress in the adhesive, respectively, which are assumed constant through the thickness of the adhesive, and  $N_i$ ,  $V_i$ , and  $M_i$  are the stress resultants for adherend *i*. Subtracting Equations (1a) gives:

\*The differential equations governing the behavior of a scarf joint are presented in Appendix A. No solution was obtained, however.

Adherend 2  

$$M_{1} = \begin{pmatrix} M_{1} \\ M_{1}$$

a) Force Free Body

Adhere

Adhesi



## FIGURE 2. FORCES AND DISPLACEMENTS FOR AN ELEMENT OF INFINITESIMAL LENGTH OF A SINGLE LAP BONDED JOINT

$$\tau = \frac{1}{2} \phi' \tag{2}$$

where

$$\phi = N_1 - N_2 \tag{3}$$

and the prime denotes differentiation with respect to x. Adding Equations (1b) and solving for  $\sigma$  yields

$$\sigma = \frac{1}{2} \frac{d}{dx} \left( V_1 + V_2 \right) \tag{4}$$

or, by substitution of (1c), and (1a),

$$\sigma = \frac{1}{2} \theta^{\prime\prime} \tag{5}$$

where

$$\theta = M_1 + M_2 + \frac{t_1 - t_2}{4} \left( N_1 - N_2 \right)$$
(6)

Equilibrium of the overall joint, as deduced from Figure 1a, requires that

$$N_1 + N_2 - P = 0 \tag{7}$$

and, by summing moments about the centerline of the adhesive,

$$M_1 = M_2 + \frac{N_1 t_1}{2} - \frac{N_2 t_2}{2} + P\left(\frac{\overline{t_C}}{2a} - \frac{\overline{t_X}}{a} - \frac{t_1}{4} + \frac{t_2}{4}\right) = 0$$
(8)

By Equation (7), this can be written as

$$M_1 - M_2 - Q\left(x - \frac{c}{2}\right) + \frac{\overline{t}}{2}\left(N_1 - N_2\right) = 0$$
(9)

where Q is taken as

$$Q = \frac{P_{\overline{t}}}{a} \tag{10}$$

and

$$\overline{t} = \frac{t_1 + t_2}{2} \tag{11}$$

and P is the joint load per unit width of joint. The quantity t has been neglected with respect to  $t_i$  (the adherend thicknesses). Note that the "a" dimension is the distance between points of zero moment.

The variables  $\phi$  and  $\theta$  will be taken as the primary unknown functions. By solving Equations (3), (6), (7), (9), and (1c) simultaneously, the stress resultants are found in terms of  $\phi$  and  $\theta$  as:

$$N_{1} = \frac{1}{2} (P + \phi)$$

$$N_{2} = \frac{1}{2} (P - \phi)$$

$$M_{1} = \frac{1}{2} \left[ \theta + \frac{Pt}{a} \left( x - \frac{c}{2} \right) - \frac{t_{1}}{2} \phi \right]$$

$$M_{2} = \frac{1}{2} \left[ \theta - \frac{Pt}{a} \left( x - \frac{c}{2} \right) + \frac{t_{2}}{2} \phi \right]$$

$$V_{1} = \frac{1}{2} \left( \theta' + \frac{Pt}{a} \right)$$

$$V_{2} = \frac{1}{2} \left( \theta' - \frac{Pt}{a} \right)$$
(12)

The adhesive stresses  $\tau$  and  $\sigma$  are also determined from  $\phi$  and  $\theta$  by Equations (2) and (5).

#### H.2.b. Compatibility Equations

Another set of equations, namely the compatibility equations, must be brought into play. The shear strain  $\gamma$  and normal strain  $\epsilon$  which are assumed constant through the adhesive are given by

$$\gamma = \frac{\overline{u_1 - \overline{u_2}}}{t}$$

$$\epsilon = \frac{v_1 + v_2}{t}$$
(1.3)

where  $\overline{u}_1$  and  $\overline{u}_2$  are the x displacements of the upper adherend lower face and the lower adherend upper face, respectively (Fig. 2b).

$$\overline{u}_{1} = u_{1} + \frac{t_{1}}{2} v_{1}'$$

$$\overline{u}_{2} = u_{2} + \frac{t_{2}}{2} v_{2}'$$
(14)

The quantities  $u_i$  and  $v_i$  are the axial and lateral displacements of the midplane of adherend *i*, respectively, as shown in Figure 2b. The middle surface normal strain  $e_i$  and curvature  $X_i$  in the adherends are given by

$$c_i = u_i' \tag{15}$$
$$X_i = -v_i''$$

where *i* refers to adherend number.

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## II.2.c. Constitutive Equations

Constitutive equations must now be introduced to relate material deformations to stresses. The bonded joints are composed of both isotropic and orthotropic materials: isotropic and/or orthotropic adherends and an isotropic adhesive.

## II.2.c.(1) Isotropic Adherend

The adherends are assumed to be in a state of plane stress in the x-z plane (Fig. 1), i.e.,  $\sigma_y = \tau_{xy} = \tau_{yz} = 0$ . (The additional assumption of plane strain in the x-y plane, i.e.,  $\epsilon_z = \gamma_{xz} = \gamma_{yz} = 0$  will be introduced later.) Two basic theories of plasticity are available for the description of the nonlinear behavior of isotropic materials deformation (total strain) theory and flow (incremental strain) theory. Deformation theory is independent of the loading path whereas flow theory depends upon the loading path. Deformation theory will be assumed here. For an isotropic material in plane stress, the deformation theory of plasticity states that the relationship between stresses and strains in the inelastic regime is<sup>(1)\*</sup>

$$\begin{cases} \epsilon_x \\ \epsilon_z \\ \gamma_{xz} \end{cases} = \left( \frac{1}{E} \begin{bmatrix} 1 & -\nu & 0 \\ -\nu & 1 & 0 \\ 0 & 0 & 2(1+\nu) \end{bmatrix} + \frac{\overline{\epsilon_p}}{\overline{\sigma}} \begin{bmatrix} 1 & -1/2 & 0 \\ -1/2 & 1 & 0 \\ 0 & 0 & 3 \end{bmatrix} \right) \begin{pmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{pmatrix}$$
(16)

where  $\epsilon_x, \epsilon_z, \gamma_{xz}$  are the plane stress strains;  $\sigma_x, \sigma_z, \tau_{xz}$  are the plane stresses in the x-z plane of Figure 1; E, v, are elastic constants;  $\sigma$ , the equivalent stress, is<sup>(1)</sup>

$$\overline{\sigma} = (\sigma_x^2 + \sigma_z^2 - \sigma_x \sigma_z + 3\tau_{xz}^2)^{1/2}$$
(17)

and  $\overline{\epsilon}_p$  is the corresponding equivalent plastic strain. Deformation theory assumes that  $\overline{\epsilon}_p$  for the plane stress case can be obtained from the uniaxial stress-strain curve at a stress level  $\overline{\sigma}$ . (The  $\overline{\sigma}$  vs  $\overline{\epsilon}_p$  curve is identical to the  $\sigma_x$  vs  $\epsilon_{xp}$ curve for uniaxial stress.) Since the plastic strain is the difference between the total strain and the elastic component, one has

$$\overline{e}_p = \overline{e} = \frac{\overline{o}}{\overline{E}}$$
(18)

where  $\overline{\epsilon}$  is equivalent total strain at a stress level  $\overline{\sigma}$ . If the Ramberg-Osgood<sup>(2)</sup> approximation to the stress-strain curve is used, the relationship between  $\overline{\sigma}$  and  $\overline{\epsilon}$  can be expressed by:

$$\overline{\epsilon} = \frac{\overline{\sigma}}{E} + \frac{3}{7} \frac{\sigma_0}{E} \left(\frac{\overline{\sigma}}{\sigma_0}\right)^n \tag{19}$$

where  $\sigma_0$  and *n* are material constants selected such that Equation (19) fits the nonlinear portion of the uniaxial stress-strain curve. (See Discrete Element Analysis, Section III.)

By combining Equations (16) and (18), one obtains

$$\begin{cases} \epsilon_x \\ \epsilon_z \\ \gamma_{xz} \end{cases} = \frac{1}{E_x} \begin{bmatrix} 1 & -\nu_p & 0 \\ -\nu_p & 1 & 0 \\ 0 & 0 & 2(1+\nu_p) \end{bmatrix} \begin{cases} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{cases}$$
(20)

\*Raised numbers in parentheses refer to entries in the List of References,

in which  $E_s$ , the secant modulus, is

$$E_s = \frac{\overline{\sigma}}{\overline{\epsilon}} \tag{21}$$

and  $v_p$ , the plastic Poisson's ratio, is

$$\nu_{p} = \frac{1}{2} \left[ 1 - \frac{E_{s}}{E} \left( 1 - 2\nu \right) \right]$$
(22)

Inversion of Equation (20) yields the stresses in terms of strain as

$$\begin{cases} \sigma_{\mathbf{x}} \\ \sigma_{\mathbf{z}} \\ \tau_{\mathbf{x}\mathbf{z}} \end{cases} = \frac{E_s}{(1-\nu_p^2)} \begin{bmatrix} 1 & \nu_p & 0 \\ \nu_p & 1 & 0 \\ 0 & 0 & \frac{(1-\nu_p)}{2} \end{bmatrix} \begin{cases} \epsilon_x \\ \epsilon_z \\ \gamma_{\mathbf{x}\mathbf{z}} \end{cases}$$
(23)

in which, now, the equivalent stress  $\overline{\sigma}$  is found from the uniaxial stress-strain curve at an equivalent strain of

$$\overline{\epsilon} = \frac{1}{(1-\nu_p^2)} \left[ (1-\nu_p + \nu_p^2)(\epsilon_x^2 + \epsilon_z^2) + (4\nu_p - 1 - \nu_p^2) \epsilon_x \epsilon_z + \frac{3}{4} (1-\nu_p)^2 \gamma_{xz}^2 \right]^{1/2}$$
(24)

which applies to the plane stress case in the x-z plane. The additional assumption of zero strain in the z direction can now be conveniently introduced ( $\epsilon_z = \gamma_{xz} = 0$ ) so that Equations (23) and (24) specialize to

$$\sigma_{x} = \frac{E_{s}}{(1 - \nu_{p}^{2})} \epsilon_{x}$$

$$\sigma_{z} = \nu_{p} \frac{E_{s}}{(1 - \nu_{p}^{2})} \epsilon_{x}$$

$$\tau_{xz} = 0$$
(25)

and

$$\overline{\epsilon} = \frac{(1 - \nu_p + \nu_p^2)^{1/2}}{(1 - \nu_p^2)} \epsilon_x$$
(26)

It will be convenient, for later developments, to separate the stress into two components - the stresses which would be present if the strains were totally elastic and the stresses which must be subtracted from these stresses to account for plasticity, i.e.,

$$\sigma_x = \frac{E}{(1 - \nu^2)} \epsilon_x - \sigma_{xp}$$

$$\sigma_z = \frac{\nu E}{(1 - \nu^2)} \epsilon_x - \sigma_{zp}$$
(27)

where the fictitious stresses  $\sigma_{xp}$  and  $\sigma_{zp}$ , herein termed plastic stresses, are given by:

$$\sigma_{xp} = \left[\frac{E}{(1-\nu^2)} - \frac{E_s}{(1-\nu_p^2)}\right] \epsilon_x = \frac{E}{(1-\nu^2)} \left[1-\eta\right] \epsilon_x$$

$$\sigma_{zp} = \left[\frac{\nu E}{(1-\nu^2)} - \frac{\nu_p E_s}{(1-\nu_p^2)}\right] \epsilon_x = \frac{\nu E}{(1-\nu^2)} \left[1 - \frac{\nu_p}{\nu_p}\eta\right] \epsilon_x$$
(28)

,

in which:

 $\eta = \frac{E_s(1 - \nu^2)}{E(1 - \nu_p^2)}$ (29)

In the analysis of the adherend, the relationship between the stress resultant in the x direction and middle surface strains is desired. In the customary fashion, one defines the stress-resultant normal force in the x direction of the adherend as:

$$V = \int_{t/2}^{t/2} \sigma_x \, \mathrm{d}y \tag{30}$$

and the stress resultant moment as

$$M = \int_{-t/2}^{t/2} \sigma_x y \, \mathrm{d}y \tag{31}$$

where t is the adherend thickness. Employing the assumption that plane sections remain plane, one has

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$$f_X = e + yX \tag{32}$$

where e is the middle surface normal strain and X is the middle surface curvature.

For an isotropic plate, one employs Equations (27), (30), (31), and (32) to arrive at the following plate constitutive equations:

$$N = Ae - N_p$$

$$M = DX - M_p$$
(33)

where

 $A = \frac{Et}{(1 - \nu^{2})}$   $D = \frac{Et^{3}}{12(1 - \nu^{2})}$ (34)

and the plastic stress resultants are:

$$N_p = \int_{-t/2}^{t/2} \sigma_{xp} \, \mathrm{d}y$$

$$M_p = \int_{-t/2}^{t/2} \sigma_{xp} y \, \mathrm{d}y$$
(35)

For purposes of numerical integration, the isotropic plate thickness will be divided into nine equal layers so that the plastic stress resultants can be written as

$$N_{p} = \frac{t}{9} \sum_{k=1}^{9} \sigma_{xp}^{k}$$

$$M_{p} = \frac{t}{9} \sum_{k=1}^{9} \sigma_{xp}^{k} v^{k}$$
(36)

where k refers to the layer number and  $y^k$  is the distance to the center of the layer f. on the middle surface. Introducing the strain-displacement relationships of Equation (15), Equation (33) can be written as

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$$u'_{i} = \frac{N_{i} + N_{ip}}{A_{i}}$$

$$(37)$$

$$''_{i} = -\frac{M_{i} + M_{ip}}{D_{i}}$$

where the subscript *i* has been added to denote I dherend number.

II.2.c.(2) Orthotropic Adherend

An appropriate modification of the deformation theory of plasticity for orthotropic materials which has been suggested by Reference (1) is a generalization of Equation (16) as

$$\begin{pmatrix} \epsilon_{\varrho} \\ \epsilon_{t} \\ \gamma_{\varrho t} \end{pmatrix} = \begin{pmatrix} \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} + \frac{\overline{\epsilon}_{p}}{\overline{\sigma}} \begin{bmatrix} \alpha_{11} & \alpha_{12} & 0 \\ \alpha_{12} & \alpha_{22} & 0 \\ 0 & 0 & 3\alpha_{66} \end{bmatrix} \end{pmatrix} \begin{pmatrix} a_{1} \\ a_{1} \end{pmatrix}$$
(38)

where the subscripts  $\ell$  and t, correspond to the principal material directions. The  $\ell$ -axis is along the fibers and the t-axis is perpendicular to the fibers in the plane of a typical lamina (see Fig. 3). The quantities  $S_{11}$ ,  $S_{12}$ ,  $S_{22}$ , and  $S_{6,6}$  are elastic constants in the principal material directions:

$$S_{11} = 1/E_{\varrho}$$

$$S_{12} = -\nu_{\varrho t}/E_{\varrho} = -\nu_{t\varrho}/E_{t}$$

$$S_{22} = 1/E_{t}$$

$$S_{66} = 1/G_{\varrho t}$$
(39)

where  $E_{\ell}$ ,  $E_t$ , and  $G_{\ell t}$  are the elastic orthotropic moduli and  $\nu_{\ell t}$  is the orthotropic Poisson's ratio. The equivalent stress corresponding to Equation (38) is:

$$\overline{\sigma} = (\alpha_{11}\sigma_{\ell}^2 + \alpha_{22}\sigma_{\ell}^2 + 2\alpha_{12}\sigma_{\ell}\sigma_{\ell} + 3\alpha_{66}\tau_{\ell}^2)^{1/2}$$
(40)

The quantities  $\alpha_{ij}$  are, in general, variables dependent upon the state of stress. Their values will be discussed shortly. It is apparent that, for isotropic materials (isotropic strain-hardening), one has  $\alpha_{11} = \alpha_{22} = \alpha_{66} = 1$  and  $\alpha_{12} = -1/2$ ; i.e., the  $\alpha_{ij}$  are constant, and  $\overline{\epsilon}_p$  is determined from a single uniaxial stress-strain curve. This will not be the case for orthotropic materials.



The values of  $\alpha_{1,1}$ ,  $\alpha_{1,2}$ ,  $\alpha_{2,2}$ ,  $\alpha_{6,6}$ , and  $\overline{e}_p$  are determined such that Equations (38) and (40) are satisfied for the conditions of uniaxial (normal and shear) stress in the principal directions. Suppose that the orthotropic material is characterized by the four uniaxial curves shown in Figure 3, which are obtained by uniaxial tests of : typical lamina. i.e., uniaxial normal stress tests in the  $\ell$  and t directions and a pure shear stress test in the  $\ell$ t-plane. Each curve is to be approximated by a Ramberg-Osgood law, so that, for the uniaxial tests, one has

- Uniaxial Stress σ<sub>ℓ</sub>

$$\epsilon_{\mathcal{K}} = S_{1,1} \sigma_{\mathcal{K}} + \frac{3}{7} S_{1,1} \sigma_{0,\mathcal{K}} \left( \frac{\sigma_{\mathcal{K}}}{\sigma_{0,\mathcal{K}}} \right)^{n_{\mathcal{K}}}$$
(41a)

$$\epsilon_t = S_{12}\sigma_{\xi} + \frac{3}{7}S_{12}\sigma_{0\xi t} \left(\frac{\sigma_{\xi}}{\sigma_{0\xi t}}\right)^{n_{\xi t}}$$
(41b)

Uniaxial Stress  $\sigma_{f}$ 

$$\epsilon_{I} = S_{22} \sigma_{I} + \frac{3}{7} S_{22} \sigma_{0I} \left(\frac{\sigma_{I}}{\sigma_{0I}}\right)^{n_{I}} \quad (41c)$$

In-Plane Shear  $\tau_{M}$ 

$$\gamma_{CI} = S_{-6} \tau_{CI} + \frac{3}{7} S_{66} \tau_{0CI} \left(\frac{\tau_{CI}}{\tau_{0CI}}\right)^{m_{CI}} (41d)$$

FIGURE 3. MATERIAL COORDINATES & t.y AND STRESS-STRAIN CURVES FROM UNIAXIAL STRESS TESTS IN PRINCIPAL DIRECTIONS

where the  $n \le m_{XI}$ , and  $\sigma_0$ 's, and  $\tau_{0XI}$  are material constants selected such that Equations (41a), (41b), (41c), and (41d) fit the curves in Figure 3. The elastic constants are the same as those given in Equation (38).

For a uniaxial test in the & direction, Equations (38) and (40) give the strain in the & direction as

$$\epsilon_{\mathcal{K}} = S_{1,1} \sigma_{\mathcal{K}} + \sqrt{\alpha_{1,1}} \,\overline{\epsilon_p} \quad ; \quad \sigma_{\mathcal{K}} = \frac{\overline{\sigma}}{\sqrt{\alpha_{1,1}}} \tag{42}$$

By comparison with Equation (41a), it is apparent that both  $\alpha_{11}$  and  $\overline{\epsilon}_p$  cannot be determined uniquely; i.e., either  $\alpha_{11}$  or  $\overline{\epsilon}_p$  is arbitrary. It is convenient to select

$$\alpha_{1,1} = 1 \tag{43}$$

so that, from (41a) and (42),

$$\overline{\epsilon}_{p} = \frac{3}{7} S_{11} \sigma_{0\ell} \left( \frac{\overline{\sigma}}{\sigma_{0\ell}} \right)^{n_{\ell}}$$
(44)

In other words, we have defined the uniaxial stress-strain curve in the  $\ell$  direction [Eq. (41a)] to be the equivalent stress-strain curve,

$$\overline{\epsilon} = S_{11}\overline{\sigma} + \frac{3}{7}S_{11}\sigma_{0\ell}\left(\frac{\overline{\sigma}}{\sigma_{0\ell}}\right)^{n_{\ell}}$$
(45)

Using the remaining three equations (41b, c, d) in a similar manner, one finds [note that  $\overline{\epsilon}_p$  is now defined by Equation (44)]

$$\alpha_{12} = \frac{S_{12}\sigma_{0\ell}}{S_{11}\sigma_{0\ell}} \left(\frac{\overline{\sigma}}{\sigma_{0\ell}}\right)^{n_{\ell}} \left(\frac{\sigma_{0\ell}}{\overline{\sigma}}\right)^{n_{\ell}} \left(\frac{\sigma_{0\ell}}{\overline{\sigma}}\right)^{n_{\ell}}$$

$$\alpha_{22} = \left[\frac{S_{22}\sigma_{0\ell}}{S_{11}\sigma_{0\ell}} \left(\frac{\overline{\sigma}}{\sigma_{0\ell}}\right)^{n_{\ell}} \left(\frac{\sigma_{0\ell}}{\overline{\sigma}}\right)^{n_{\ell}}\right]^{\frac{2}{n_{\ell}+\ell}} (46)$$

$$\alpha_{66} = \frac{1}{3} \left[ \frac{S_{66} \tau_{0\xi_{l}}}{S_{11} \sigma_{0\xi_{l}}} \left( \frac{\overline{\sigma}}{\tau_{0\xi_{l}}} \right)^{m_{\xi_{l}}} \left( \frac{\sigma_{0\xi}}{\overline{\sigma}} \right)^{n_{\xi_{l}}} \right]^{\frac{2}{m_{\xi_{l}} + 1}}$$

Replacing  $\overline{\epsilon}_p$  by

$$\overline{c}_p = \overline{c} + S_{11}\overline{a} \tag{47}$$

in Equation (38), and letting  $\alpha_{11}$  be unity in Equation (40), gives

$$\begin{pmatrix} \epsilon_{\zeta} \\ \cdot \\ \cdot \\ \gamma_{\zeta T} \end{pmatrix} = \begin{bmatrix} S_{11\zeta} & S_{12\zeta} & 0 \\ S_{12\zeta} & S_{22\zeta} & 0 \\ 0 & 0 & S_{65\zeta} \end{bmatrix} \begin{pmatrix} \sigma_{\zeta} \\ \cdot \\ \sigma_{T} \\ \cdot \\ \tau_{\zeta T} \end{pmatrix}$$
(48)

and

$$\overline{\sigma} = (\sigma_1^2 + \alpha_{2,2}\sigma_1^2 + 2\alpha_{1,2}\sigma_1\sigma_1 + 3\alpha_{6,6}\tau_{(1)}^2)^{1/2}$$
(49)

where the orthotropic secant compliance elements are given by:

$$S_{11s} = \frac{\overline{\epsilon}}{\overline{\sigma}}$$

$$S_{12s} = S_{12} + \alpha_{12}(S_{11s} - S_{11})$$

$$S_{22s} = S_{22} + \alpha_{22}(S_{11s} - S_{11})$$

$$S_{60s} = S_{6p} + 3\alpha_{60}(S_{11s} - S_{11})$$
(50)

which corresponds to Equation (20) for isotropic materials. For a given stress state  $\sigma_{\zeta}$ ,  $\sigma_{\ell}$ ,  $\tau_{\zeta\ell}$  and given material constants, Equations (45), (46), (48), and (49) completely define the state of strain. However, the definition is not explicit since Equations (45), (46), and (49) represent five nonlinear equations in the five unknowns  $\bar{\sigma}$ ,  $\bar{\epsilon}$ ,  $\alpha_{12}$ ,  $\alpha_{22}$ , and  $\alpha_{66}$ .

Inversion of Equation (48) yields the stresses in terms of the strains as

$$\begin{cases} \sigma_{\varrho} \\ \sigma_{t} \\ \tau_{\varrho t} \end{cases} = \begin{bmatrix} Q_{11s} & Q_{12s} & 0 \\ Q_{12s} & Q_{22s} & 0 \\ 0 & 0 & Q_{66s} \end{bmatrix} \begin{pmatrix} \epsilon_{\varrho} \\ \epsilon_{t} \\ \gamma_{\varrho t} \end{pmatrix}$$
(51)

where the secant stiffness elements  $\mathcal{Q}_{ijs}$ , are given by

$$Q_{11s} = S_{22s} / (S_{11s} S_{22s} - S_{12s}^2)$$

$$Q_{12s} = -S_{12s} / (S_{11s} S_{22s} - S_{12s}^2)$$

$$Q_{22s} = S_{11s} / (S_{11s} S_{22s} - S_{12s}^2)$$

$$Q_{66s} = 1/S_{66s}$$
(52)

The equivalent strain, obtained by combining Equations (50), (49) and (51), is found as:

$$\overline{\epsilon} = (\beta_{11}\epsilon_{\ell}^2 + \beta_{22}\epsilon_{\ell}^2 + 2\beta_{12}\epsilon_{\ell}\epsilon_{\ell} + 3\beta_{66}\gamma_{\ell \ell}^2)^{1/2}$$
(53)

in which

$$\beta_{11} = (Q_{11s}^2 + \alpha_{22}Q_{12s}^2 + 2\alpha_{12}Q_{11s}Q_{12s})S_{11s}^2$$

$$\beta_{12} = [Q_{11s}Q_{12s} + \alpha_{22}Q_{22s}Q_{12s} + \alpha_{12}(Q_{11s}Q_{22s} + Q_{12s}^2)]S_{11s}^2$$

$$\beta_{22} = (Q_{12s}^2 + \alpha_{22}Q_{22s}^2 + 2\alpha_{12}Q_{1-s}Q_{22s})S_{11s}^2$$

$$\beta_{66} = \alpha_{66}Q_{66s}^2S_{11s}^2$$
(54)

Separating the stresses into two components, as for isotropic materials in Equation (27), gives

$$\begin{pmatrix} \sigma_{\varrho} \\ \sigma_{\ell} \\ \tau_{\varrho_{\ell}} \end{pmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{pmatrix} \epsilon_{\varrho} \\ \epsilon_{\ell} \\ \gamma_{\varrho_{\ell}} \end{pmatrix} = \begin{pmatrix} \sigma_{\varrho_{p}} \\ \sigma_{\ell p} \\ \tau_{\varsigma_{\ell p}} \end{pmatrix}$$
(55)

in which the elastic stiffness elements  $Q_{ij}$  are as given in Equation (52) without the subscript s, i.e., in terms of the elastic compliance elements in Equation (39), and

$$\begin{pmatrix} \sigma_{\ell p} \\ \sigma_{\ell p} \\ \sigma_{\ell p} \\ \tau_{\varrho \ell p} \end{pmatrix} = \begin{bmatrix} Q_{11} - Q_{11s} & Q_{12} - Q_{12s} & 0 \\ Q_{12} - Q_{12s} & Q_{22} - Q_{22s} & 0 \\ 0 & 0 & Q_{66} - Q_{66s} \end{bmatrix} \begin{pmatrix} \epsilon_{\ell} \\ \epsilon_{\ell} \\ \gamma_{\varrho_{\ell}} \end{pmatrix}$$
(56)

When dealing with orthotropic materials, it is necessary to transform stresses and strains from the x-z coordinate system to the principal material directions ( $\ell$ -t) and vice versa (see Fig. 3). The stresses in the x-z coordinate system are given in terms of the stresses in the  $\ell$ -t coordinate system by

$$\begin{cases} \sigma_{x} \\ \sigma_{z} \\ \tau_{xz} \end{cases} = \begin{bmatrix} c^{2} & s^{2} & -2sc \\ s^{2} & c^{2} & 2sc \\ sc & -sc & c^{2} - s^{2} \end{bmatrix} \begin{pmatrix} \sigma_{\varrho} \\ \sigma_{f} \\ \tau_{\varrho_{f}} \end{pmatrix}$$
(57)

where

$$s = \sin \psi$$
 (58)  
 $y = \cos \psi$ 

and  $\psi$  is shown in Figure 3. Similarly, the strains in the  $\ell$ -*t* plane, due to strains in the *x* direction, are (note, by the plane strain assumption in the *x*-*y* plane,  $\epsilon_z = \gamma_{xz} = 0$ )

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$$\begin{cases} \epsilon_{\varrho} \\ \epsilon_{I} \\ \gamma_{\varrho I} \end{cases} = \epsilon_{\chi} \begin{cases} c^{2} \\ s^{2} \\ -2cs \end{cases}$$
(59)

In a manner similar to that used for isotropic adherends, Equations (55), (30), (31), and (32) and the transformation Equation (57) are used to obtain the constitutive equations for a laminated plate composed of different layers of an orthotropic material. Assuming symmetry of the laminated adherend plate about its midplane, one again arrives at Equation (33) for the plate constitutive equations, except in this case the constants A and D are defined as:

$$A = \sum_{k} \bar{Q}_{11}^{k} t^{k}$$
$$D = \sum_{k} \bar{Q}_{11}^{k} \left[ t^{k} (v^{k})^{2} + \frac{(t^{k})^{3}}{12} \right]$$

where  $t^k$  is the thickness of layer k,

$$\overline{Q}_{11}^{k} = [Q_{11}c^{4} + 2(Q_{12} + 2Q_{66})c^{2}s^{2} + Q_{22}s^{4}]_{k}$$
(61)

and s and c are defined in Equation (58). In Equation (60), the sum on k is taken over all the layers in the laminate. The plastic stress resultants for the laminate are found as

$$N_p = \sum_{k} \sigma_{xp}^{k} t^{k}$$

$$M_p = \sum_{k} \sigma_{xp}^{k} t^{k} y^{k}$$
(62)

(60)

#### II.2.c.(3) Isotropic Adhesive

The adhesive is ssumed to be an isotropic material and the constitutive equations of Reference (1) are utilized. The relations for the three-dimensional stress state will now be specialized for the adhesive. The normal stress in the x direction is neglected,  $\sigma_x = 0$ . Since the joint is assumed to be in a state of plane strain in the x-y plane, one has  $\gamma_{yz} = \gamma_{yz} = 0$ . Hence, the equations of Reference (1) apply to the adhesive in the following form

where now

$$\overline{\sigma} = (\sigma_V^2 + \sigma_Z^2 - \sigma_V \sigma_Z + 3\tau_{XV}^2)^{1/2}$$
(64)

is the equivalent stress.  $E_s$  and  $v_p$  are defined by Equations (21) and (22) respectively. Inversion of Equation (63) and introduction of the other zero strain for the plane strain assumption,  $\epsilon_z = 0$ , gives

 $\sigma_{y} = \frac{E_{s}}{(1 - \nu_{p}^{2})} \epsilon_{y}$   $\sigma_{z} = \frac{\nu_{p} E_{s}}{(1 - \nu_{p}^{2})} \epsilon_{y}$   $\tau_{xy} = \frac{E_{s}}{2(1 + \nu_{p})} \gamma_{xy}$ (65)

Now the equivalent stress  $\overline{\sigma}$  is found from the uniaxial stress strain curve at an equivalent strain of

$$\overline{\epsilon} = \frac{1}{(1 - \nu_p^2)} \left[ (1 - \nu_p + \nu_p^2) \epsilon_y^2 + \frac{3}{4} (1 - \nu_p)^2 \gamma_{xy}^2 \right]^{1/2}$$
(66)

Separating the elastic and inelastic portions of the stress, as was done for the adherends, one finds the total stresses

$$\sigma = \frac{E\epsilon}{(1 - \nu^2)} - \sigma_p$$

$$\tau = G\gamma - \tau_p$$
(67)

in which the subscripts on the adhesive stresses and strains have been removed, i.e.,  $\sigma$  has replaced  $\sigma_y$ ,  $\tau$  replaced  $\tau_{xy}$ ,  $\epsilon$  replaced  $\epsilon_y$ , and  $\gamma$  replaced  $\gamma_{xy}$ . The plastic stresses are given by

$$a_p = \left[\frac{E}{(1-\nu^2)} - \frac{E_s}{(1-\nu_p^2)}\right]\epsilon$$

$$\tau_p = \left[G - \frac{E_s}{2(1+\nu_p)}\right]\gamma$$
(68)

Introducing the compatibility equations for the adhesive [Equation (13)], the stress-displacement relationship for the adhesive is

$$\tau = \frac{G}{t} (u_1 - u_2) + \frac{G}{2t} (t_1 v_1' - t_2 v_2') - \tau_p \tag{69a}$$

$$\sigma = \frac{E}{(1 - \nu^2)t} (\nu_1 + \nu_2) - \sigma_p \tag{69b}$$

The constitutive equations for the adhesive and adherends are now complete.

## 11.2.d. Governing Differential Equation

The equilibrium, compatibility, and constitutive equations are combined to develop the governing differential equations for the single lap joint. In particular, by substituting Equations (2), (5), and (37) into (69) and employing (12) gives

$$\theta^{\prime\prime\prime\prime} + p_1 \theta - p_2 \phi = q_1 + q_2^{\prime\prime}$$

$$\phi^{\prime\prime} - p_3 \phi + p_4 \theta = q_3 + q_4^{\prime}$$
(70)

where

$$p_{1} = \frac{E}{t(1-v^{2})} \left( \frac{1}{D_{1}} + \frac{1}{D_{2}} \right)$$

$$p_{2} = \frac{F}{2t(1-v^{2})} \left( \frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}} \right)$$

$$p_{3} = \frac{G}{t} \left( \frac{1}{A_{1}} + \frac{1}{A_{2}} + \frac{t_{1}^{2}}{4D_{1}} + \frac{t_{2}^{2}}{4D_{2}} \right)$$

$$p_{4} = \frac{G}{t} \left( \frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}} \right)$$

$$q_{1} = -\frac{PE\overline{t}}{at(1-v^{2})} \left( \frac{1}{D_{1}} - \frac{1}{D_{2}} \right) \left( x - \frac{c}{2} \right) - \frac{2E}{t(1-v^{2})} \left( \frac{M_{1p}}{D_{1}} + \frac{M_{2p}}{D_{2}} \right)$$

$$q_{2} = -2\sigma_{p}$$

$$q_{3} = \frac{PG}{t} \left[ \frac{1}{A_{1}} - \frac{1}{A_{2}} + \frac{\overline{t}}{2a} \left( \frac{t_{1}}{D_{1}} + \frac{t_{2}}{D_{2}} \right) \left( \frac{c}{2} - x \right) \right] + \frac{G}{t} \left[ \frac{2N_{1p}}{A_{1}} - \frac{2N_{2p}}{A_{2}} - \frac{t_{1}M_{1p}}{D_{1}} + \frac{t_{2}M_{2p}}{D_{2}} \right]$$

$$q_{4} = -2\tau_{p}$$
(71)

Equations (70) represent the governing differential equations for the single lap joint. It will be noted that they are nonlinear equations since the plastic quantities  $N_{ip}$ ,  $M_{ip}$ ,  $\sigma_p$ , and  $\tau_p$  are nonlinear functions of the displacements. However, as the equations are written, the portions on the left are linear differential equations with constant coefficients ( $p_i$  are constants). The portions on the right ( $q_i$ ) which contain the plastic portions are nonlinear. Equations (70) are, thus, in proper form for an iterative solution to be discussed in Section II.5. It is apparent that the equations become uncoupled if both adherends are identical; i.e.,  $p_2 = p_4 = 0$ .

From Figure 1a, the boundary conditions for the single lap joint are developed by requiring that the stress resultants in the upper adherend must be zero at x = 0 and the stress resultants in the lower adherend be zero at x = c:

$$N_1 = M_1 = V_1 = 0$$
,  $x = 0$   
 $N_2 = M_2 = V_2 = 0$  at  $x = c$ 
(72)

By substituting Equations (12) into (72), one arrives at the boundary conditions in terms of  $\phi$  and  $\theta$  as

$$\frac{x=0}{\phi_0 = -P} \qquad \qquad \frac{x=c}{\phi_c = P}$$

$$\theta_0 = \frac{P}{2} \left( \frac{c\overline{t}}{a} - t_1 \right) \qquad \qquad \theta_c = \frac{P}{2} \left( \frac{c\overline{t}}{a} - t_2 \right) \qquad (73)$$

$$\theta_0' = -\frac{P\overline{t}}{a} \qquad \qquad \theta_c' = \frac{P\overline{t}}{a}$$

These be undary conditions, along with Equations (70) and the constitutive equations of Section II.2.c, are the governing equations for the bonded single lap joint.

## **II.3. DOUBLE LAP JOINT**

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The free body in Figure 4 is used to develop the governing equations for the bonded double lap joint. Since the joint is assumed to be symmetrical about the midplane of adherend 2, the moment  $M_2$  and shear  $V_2$  vanish everywhere. Equating the total forces and



where the notation is as defined previously in accordance with Figures 1 and 4. Subtracting

$$\tau = \frac{1}{2}\phi' \tag{75}$$

(74a)

(74b)

(74c)

where

 $\phi = N_{11} - \frac{N_2}{2}$ (76) The normal stress is given by the first of Equations (74b) as

$$\sigma = \frac{\mathrm{d}V_1}{\mathrm{d}x} \tag{77}$$

or, introducing Equations (74a) and (74c),

$$\sigma = \frac{1}{2}\theta^{\prime\prime} \tag{78}$$

where

$$\theta = 2 \left[ M_1 + \frac{t_1}{4} \left( N_1 - \frac{N_2}{2} \right) \right]$$
(79)

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Equilibrium of the overall joint requires that (see Fig. 1b)

$$N_1 + \frac{N_2}{2} - P = 0$$
 (9)

Solving for the stress resultants from Equations (74c), (76), (79), and (80) gives the stress resultants in terms of the unknown functions  $\phi$  and  $\theta$  as

$$N_{1} = \frac{1}{2} (P + \phi)$$

$$N_{2} = P - \phi$$

$$M_{1} = \frac{1}{2} \left( \theta - \frac{t_{1}}{2} \phi \right)$$

$$V_{1} = \frac{\theta'}{2}$$
(81)

One now proceeds to introduce the compatibility and constitutive equations, (69) and (37), in a manner almost identical to that followed for the single lap joint. Noting that, by symmetry, the lateral displacement of Adherend 2 is zero, one finds the governing differential equations to be:

$$\theta''' + p_1 \theta - p_2 \phi = q_1 + q_2''$$

$$\phi'' - p_3 \phi + p_4 \theta = q_3 + q_4'$$
(82)

where now the quantities  $p_i$  and  $q_i$  are defined by

 $q_3$ 

$$p_{1} = \frac{E}{(1 - \nu^{2})tD_{1}} \qquad p_{2} = \frac{Et_{1}}{2(1 - \nu^{2})tD_{1}}$$

$$p_{3} = \frac{G}{t} \left(\frac{1}{A_{1}} + \frac{2}{A_{2}} + \frac{t_{1}^{2}}{4D_{1}}\right) \qquad p_{4} = \frac{Gt_{1}}{2tD_{1}}$$

$$q_{1} = -\frac{2EM_{1p}}{tD_{1}(1 - \nu^{2})} \qquad q_{2} = -2\sigma_{p}$$

$$= \frac{PG}{t} \left(\frac{1}{A_{1}} - \frac{2}{A_{2}}\right) + \frac{G}{t} \left(\frac{2N_{1p}}{A_{1}} - \frac{2N_{2p}}{A_{2}} - \frac{t_{1}M_{1p}}{D_{1}}\right) \qquad q_{4} = -2\tau_{p}$$
(83)

The reader will note the obvious similarity between these equations and those for the single lap joints. In fact, by teplacing  $N_2$  with  $N_2/2$ ,  $A_2$  with  $A_2/2$ , and  $i/D_2$  with zero, and dropping the x - c/2 term (letting  $\overline{t} = 0$ , which accounts for the shear Q) in Equations (71), one arrives at Equation (83). Similarly, one may deduce Equations (81) from (12).

The boundary conditions for the double lap are also only slightly different from those for the single lap. Equations (72) also apply to the double lap, but the equations  $M_2 = V_2 = 0$  at x = c are redundant since this has already been used as a symmetry condition. Hence, two additional equations are required at x = c. By symmetry of the joint about the vertical centerline in Figure 1, the shear  $V_1$  must be zero at x = c. It is now assumed that  $M_1$  is also zero at x = c. The boundary conditions then become, by Equations (81),

$$\frac{x=0}{\phi_0 = -P} \qquad \frac{x=c}{\phi_c = P}$$

$$\theta_0 = -\frac{Pt_1}{2} \qquad \theta_c = \frac{Pt_1}{2}$$

$$\theta'_0 = 0 \qquad \theta'_c = 0$$
(84)

The solution of the nonlinear Equation (82) with the boundary conditions (84) and the constitutive equations will be discussed in Section II.5.

#### **II.4. STEP LAP JOINT**

Figure 1c shows schematically a step lap bonded joint under an axial tension, P. The total number of horizontal sections or "treads" is R, the number of risers being R + 1. The notation remains the same as for the single and double lap, except that it now becomes necessary, at times, to identify the particular step under consideration by a subscript r. For example,  $t_{1r}$  is the thickness of the upper adherend (adherend 1 in Fig. 1c) at the rth tread. The subscript r will be omitted unless it is necessary for clarity.

The derivation of the governing differential equations for the step lap follows the development for the single lap very closely. Using the free body in Figure 2, the shear stress and normal stress in the adhesive are found as in Equations (2) and (5) to be

 $\tau = \frac{1}{2} \phi' \tag{85}$ 

where

 $\phi = N_1 - N_2 \tag{86}$ 

and

$$\sigma = \frac{1}{2} \theta^{\prime\prime} \tag{87}$$

where

$$\theta = M_1 + M_2 + \frac{t_1 - t_2}{4} \left( N_1 - N_2 \right) \tag{88}$$

Equilibrium of the overall joint, as found from Figure 1c, requires that

$$N_1 + N_2 - P = 0 \tag{89}$$

and

q

 $q_4 = -2\tau_p$ 

$$M_1 - M_2 + \frac{\overline{t}}{2}(N_1 - N_2) - \frac{P}{4}(t_1 - t_2) = 0$$
(90)

The adherend stress resultants are found in terms of the unknown functions  $\phi$  and  $\theta$  by solving Equations (86), (88), (89), and (90) simultaneously:

$$N_{1} = \frac{1}{2} (P + \phi) \qquad M_{1} = \frac{1}{2} \left[ \theta + \frac{P}{4} (t_{1} - t_{2}) - \frac{t_{1}}{2} \phi \right] \qquad V_{1} = \frac{\theta'}{2}$$

$$N_{2} = \frac{1}{2} (P - \phi) \qquad M_{2} = \frac{1}{2} \left[ \theta - \frac{P}{4} (t_{1} - t_{2}) + \frac{t_{2}}{2} \phi \right] \qquad V_{2} = \frac{\theta'}{2}$$
(91)

The constitutive and compatibility conditions, (69) and (37)\*, are now introduced in a manner similar to that used for the single lap. The governing differential equation is found to be

$$\theta^{\prime\prime\prime\prime} + p_1 \theta - p_2 \phi = q_1 + q_2^{\prime\prime}$$

$$\phi^{\prime\prime} - p_3 \phi + p_4 \theta = q_3 + q_4^{\prime}$$
(92)

in which, except for  $q_1$  and  $q_3$ , the  $p_i$  and  $q_i$  are identical to those listed for the single lap; i.e.:

$$p_{1} = \frac{E}{t(1-\nu^{2})} \left( \frac{1}{D_{1}} + \frac{1}{D_{2}} \right)$$

$$p_{2} = \frac{E}{2t(1-\nu^{2})} \left( \frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}} \right)$$

$$p_{3} = \frac{G}{t} \left( \frac{1}{A_{1}} + \frac{1}{A_{2}} + \frac{t_{1}^{2}}{4D_{1}} + \frac{t_{2}^{2}}{4D_{2}} \right)$$

$$p_{4} = \frac{G}{2t} \left( \frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}} \right)$$

$$q_{1} = -\frac{PE}{t(1-\nu^{2})} \left( \frac{t_{1}-t_{2}}{4} \right) \left( \frac{1}{D_{1}} - \frac{1}{D_{2}} \right) - \frac{2E}{t(1-\nu^{2})} \left( \frac{M_{1}p}{D_{1}} + \frac{M_{2}p}{D_{2}} \right)$$

$$q_{2} = -2\sigma_{p}$$

$$q_{3} = \frac{PG}{t} \left[ \frac{1}{A_{1}} - \frac{1}{A_{2}} - \left( \frac{t_{1}-t_{2}}{8} \right) \left( \frac{t_{1}}{D_{1}} + \frac{t_{2}}{D_{2}} \right) \right] + \frac{G}{t} \left[ \frac{2N_{1}p}{A_{1}} - \frac{2N_{2}p}{A_{2}} - \frac{t_{1}M_{1}p}{D_{1}} + \frac{t_{2}M_{2}p}{D_{2}} \right]$$
(93)

\*By use of Equation (37), it has been tacitly assumed that the adherends are symmetrical at each step. This may not be the typical case for the real problem but, considering the other assumptions involved in this analysis, it will be assumed that the stretch/bending coupling induced by asymmetry can be neglected.
The reader is reminded that Equation (92) applies only for one tread of the step lap, and that, in fact, each quantity should have a subscript r to refer it to the rth tread.

When Equation (92) has been written and its general solution found for each of the R treads, the individual solutions must be connected by a set of continuity conditions at each riser. The general solution of Equations (92) for each tread has six arbitrary constants or a total of 6R constants if there are R steps. Thus, we need six continuity conditions at each intermediate riser, giving 6(R - 1) equations. With three boundary conditions at each end riser, there are 6R equations.

The continuity and boundary conditions will now be derived by the use of Figure 5 which shows a segment of the joint at riser number r. The length of the segment,  $\delta$ , goes to zero in the limit. The continuity conditions for the



FIGURE 5. FORCES AND DISPLACEMENTS OF rth RISER

stress resultants are obtained by satisfying equilibrium of the free body of the upper adherend in Figure 5a. Assuming that no force is transmitted by the adhesive in the riser, one has

$$(M_1)_{r-1} - M_{1r} + N_{1r} \left[ \frac{t_{1r}}{2} - \frac{(t_1)_{r-1}}{2} \right] = 0$$
(94a)

$$(N_1)_{r-1} - N_{1r} = 0 \tag{94b}$$

$$(V_1)_{r-1} - V_{1r} = 0 \tag{94c}$$

where now the subscripts r and r-1 are required to refer to a particular tread. According to Figure 1c, the quantities for tread r-1 are evaluated at  $x_{r-1} = b_{r-1}$  and for tread r at  $x_r = 0$ .

From Figure 5b, continuity of the lateral displacements and rotations requires that

$$(v_1)_{r-1} = v_{1r} \qquad (v_2)_{r-1} = v_{2r} \tag{95a}$$

$$(v'_1)_{r-1} = v'_{1r}$$
  $(v'_2)_{r-1} = v'_{2r}$  (95b)

Continuity of longitudinal displacements is satisfied if

$$u_{1r} = (u_1)_{r-1} + \frac{h}{2} (v'_1)_{r-1}$$

$$u_{2r} = (u_2)_{r-1} - \frac{h}{2} (v'_2)_{r-1}$$
(95c)

where

$$h = t_{1r} - (t_1)_{r-1} = (t_2)_{r-1} - t_{2r}$$

By subtracting Equations (95c), we obtain

$$u_{1r} - u_{2r} = (u_1)_{r-1} - (u_2)_{r-1} + \frac{h}{2} \left[ (v_1')_{r-1} + (v_2')_{r-1} \right]$$
(96)

Now, from Equation (69a), one finds

$$u_{1r} - u_{2r} = \frac{t}{G} \left( \tau_r + \tau_{pr} \right) - \frac{1}{2} \left( t_{1r} v_{1r}' - t_{2r} v_{2r}' \right)$$
(97)

and, similarly, for tread r = 1. Introducing Equation (97) for treads r and r = 1 into Equation (96) and using Equation (95b) gives

$$\tau_r + \tau_{pr} - \frac{Gh}{2t} (v'_{1r} + v'_{2r}) = \tau_{r-1} + (\tau_p)_{r-1} + \frac{Gh}{2t} [(v'_{1})_{r-1} + (v'_{2})_{r-1}]$$
(98)

By Equation (69b), this becomes

$$\tau_r + \tau_{pr} - \frac{Gh(1-\nu^2)}{2E} (\sigma'_r + \sigma'_{pr}) = \tau_{r-1} + (\tau_p)_{r-1} + \frac{Gh(1-\nu^2)}{2E} [\sigma'_{r-1} + (\sigma'_p)_{r-1}]$$
(99)

which represents the continuity equation for longitudinal displacements. The continuity equation for lateral displacements is obtained by adding Equations (95a) to obtain:

$$(v_1)_{r-1} + (v_2)_{r-1} = v_{1r} + v_{2r}$$
(100)

which, by Equation (69b), becomes

$$\sigma_{r-1} + (\sigma_p)_{r-1} = \sigma_r + \sigma_{pr} \tag{101}$$

Similarly, the equation for continuity of rotations becomes

$$\sigma'_{r-1} + (\sigma'_p)_{r-1} = \sigma'_r + \sigma'_{pr}$$
(102)

Equations (94a), (94b), (94c), (99), (101), and (102) are the six required continuity conditions to be employed at each riser. By introducing Equations (85), (87), and (91) and the definitions of  $q_i$  in Equations (93), these continuity conditions become, respectively,

$$\begin{aligned}
\theta_{r-1} + \frac{n}{2} \phi_{r-1} &= \theta_r \\
\phi_{r-1} &= \phi_r \\
\theta_{r-1}' &= \theta_r' \\
\theta_{r-1}' &= \theta_r' \\
\phi_{r-1}' &= \theta_r'' \\
\phi_{r-1}' &= \theta_r'' \\
\frac{Gh(1 - \nu^2)}{2E} \left[\theta_{r-1}''' - (q_2')_{r-1}\right] &= \phi_r' - q_{4r} - \frac{Gh(1 - \nu^2)}{2E} \left[\theta_{r}''' - q_{2r}'\right] \\
\theta_{r-1}'' - (q_2)_{r-1} &= \theta_r'' - q_{2r} \\
\theta_{r-1}''' - (q_2')_{r-1} &= \theta_r''' - q_{2r}'
\end{aligned}$$
(103)

where again the subscript r-1 refers to a quantity for tread r-1 evaluated at  $x_{r-1} = b_{r-1}$ , and the subscript r to a quantity for tread r evaluated at  $x_r = zero$ .

The boundary conditions at the two end risers, r = 1 and r = R + 1, are obtained from equilibrium considerations. At the first riser, one has

$$N_{11} = M_{11} = V_{11} = 0 \tag{104}$$

and, at the R + 1 riser,

$$N_{2R} = M_{2R} = V_{2R} = 0 \tag{105}$$

In terms of  $\phi$  and  $\theta$ , the boundary conditions become, respectively,

$$\frac{x_{1} = 0}{\phi_{1} = -P} \qquad \qquad \frac{x_{R} = b_{R}}{\phi_{R} = P}$$

$$\theta_{1} = \frac{P}{4}(t_{21} - 3t_{11}) \qquad \qquad \theta_{R} = \frac{P}{4}(t_{1R} - 3t_{2R})$$

$$\theta_{1}' = 0 \qquad \qquad \theta_{R}'' = 0$$
(106)

where the second subscript again refers to the tread number. The solution of the nonlinear Equations (92) with the compatibility Equations (103), boundary conditions (106), and constitutive equations for the step lap joint will be discussed in Section 11.5.

### **II.5. SOLUTION TO DIFFERENTIAL EQUATIONS**

#### II.5.a. General Procedure

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The differential equations governing the behavior of single lap (70), double lap (82), and step lap (92) bonded joints are all of the same general form:

$$0'''' + p_1 \theta - p_2 \phi = q_1 + q_2''$$

$$\phi'' - p_3 \phi + p_4 \theta = q_3 + q_4'$$
(107)

where the  $p_i$  are constants and the  $q_i$  are functions of the applied load P, the plastic stress resultants in the adherends, and the plastic stresses in the adhesive. The quantities  $p_i$  and  $q_i$  are given in Sections II.2, II.3 and II.4 for each of the respective joints. It is apparent that this set of equations is actually two nonlinear coupled differential equations since the  $q_i$  are nonlinear functions of the stress resultants and stresses and, hence, of  $\phi$  and  $\theta$ . The general approach will be to solve this set of nonlinear equations by iteration. Equation (107) is written as

$$(\theta^{\prime\prime\prime\prime})^{j} + p_{1}\theta^{j} - p_{2}\phi^{j} = q_{1}^{j-1} + (q_{2}^{\prime\prime})^{j-1}$$

$$(\phi^{\prime\prime})^{j} - p_{3}\phi^{j} + p_{4}\theta^{j} = q_{3}^{j-1} + (q_{4}^{\prime})^{j-1}$$
(108)

where j refers to the current iteration. The portions on the left-hand side of the equations are linear differential equations with constant coefficients for which a solution can be found. Thus, supposing at iteration j we have the  $q_i$  from the previous iteration (j-1), then the functions  $\theta$  and  $\phi$  for the current iteration j are obtained by solving

Equations (108) subject to the boundary (and continuity for the step lap) conditions. Symbolically, this may be written as

$$\theta^{j} = \theta^{j}(q_{i}^{j-1}, x)$$

$$\phi^{j} = \phi^{j}(q_{i}^{j-1}, x)$$
(109)

With these functions of  $\theta$  and  $\phi$ , the adhesive stresses and adherend stress resultants may be obtained by equations presented in Section II.2, II.3, or II.4, e.g., Equations (2), (5), and (12):

$$\tau^{i} = \tau^{j}(\phi^{j})$$

$$\sigma^{j} = \sigma^{j}(\theta^{j})$$

$$N_{i}^{j} = N_{i}^{j}(\theta^{j}, \phi^{j}, x)$$

$$M_{i}^{j} = M_{i}^{j}(\theta^{j}, \phi^{j}, x)$$
(110)

where *i* refers to the adherend number. Using these new values of the stresses and the previous values of the plastic stresses (contained in  $q_i^{j-1}$ ), the pertinent strains can be computed by the constitutive equations of Section II.2, e.g., Equation (20). These strains are then employed to compute new values of the plastic stress, e.g., by Equation (28), and, hence, new values of  $q_i$ :

1

$$q_{i}^{j} = q_{i}^{j}(\tau^{j}, \sigma^{j}, N_{i}^{j}, M_{j}^{j}, P, x)$$
(111)

This completes iteration j. The solution process now returns to Equation (109) to begin iteration j + 1. Iteration continues until there is an insignificant change in the plastic stresses. This solution will be discussed in more detail in the remainder of this section.

#### II.5.b. Homogeneous Solution

The homogeneous equation corresponding to Equations (107) is

$$\theta^{\prime\prime\prime\prime\prime} + p_1\theta = p_2\phi = 0 \tag{112}$$

$$\phi^{\prime\prime} - p_2\phi + p_4\theta = 0$$

The superscript j's have been eliminated here as well as in the following developments since they are not needed for clarity. The reader should remember, however, that the solution is not in closed form and that an iterative procedure is involved. Following the standard procedure for the solution of linear differential equations with constant coefficients, one assumes a solution of the form

$$\phi = C_1 e^{\rho x}$$

$$\theta = C_2 e^{\rho x}$$
(113)

Substituting Equations (113) into (112) and setting the determinate of the coefficients  $C_1$  and  $C_2$  equal to zero, one finds the characteristic equation for  $\rho$  to be

$$\Gamma^3 - p_3 \Gamma^2 + p_1 \Gamma - p_1 p_3 + p_2 p_4 = 0 \tag{114}$$

where

 $\Gamma = \rho^2 \tag{115}$ 

Equation (114) has one real, positive root,  $\Gamma_1$ , which can be found numerically. (Note that for equal adherends in the single lap,  $p_2 = p_4 = 0$ , and the real, positive root is give  $\gamma$  by  $\Gamma_1 = p_3$ .)\*

$$\Gamma_1 = \text{real, positive root of Equation (114)}$$
 (116)

Then the other two roots are the complex conjugates:

$$\Gamma_2, \Gamma_3 = \zeta_1 \pm i \zeta_2 \tag{117}$$

where

$$\zeta_{1} = \frac{p_{3} - \Gamma_{1}}{2}$$

$$\zeta_{2} = \left[\frac{p_{1}p_{3} - p_{2}p_{4}}{\Gamma_{1}} - \left(\frac{\Gamma_{1} - p_{3}}{2}\right)^{2}\right]^{1/2}$$

$$i = \sqrt{-1}$$
(118)

The complete set of six roots for  $\rho$  can now be written using Equations (115) and (117) as

$$\rho_{1} = \lambda$$

$$\rho_{2} = -\lambda$$

$$\rho_{3} = \alpha + i\beta$$

$$\rho_{4} = -\alpha - i\beta$$

$$\rho_{5} = \alpha - i\beta$$

$$\rho_{6} = -\alpha + i\beta$$

where

$$\lambda = \sqrt{\Gamma_1}$$

$$\alpha = \left[ \frac{p_3}{4} - \frac{\Gamma_1}{4} + \sqrt{\frac{p_1 p_3 - p_2 p_4}{4\Gamma_1}} \right]^{1/2}$$
(120)
$$\beta = \left[ \frac{\Gamma_1}{4} - \frac{p_3}{4} + \sqrt{\frac{p_1 p_3 - p_2 p_4}{4\Gamma_1}} \right]^{1/2}$$

\*Equation (114) will have one real root and two complex conjugate roots if

$$27 p_2^2 p_4^2 + 4p_1^3 + 4p_3^3 (p_1 p_3 - p_2 p_4) + p_1 p_3 (8p_1 p_3 - 36 p_2 p_4) > 0$$

This will generally be true since  $p_1 p_3 \gg p_2 p_4$ .

The homogeneous solution to Equation (112) can, thus, be written in matrix notation as

$$\phi = \{f(x)\}^T \quad \{C_1\}$$

$$\theta = \{f(x)\}^T \quad \{C_2\}$$
(121)

in which

$$\{f(x)\} = \begin{cases} e^{\lambda x} \\ e^{-\lambda x} \\ e^{\alpha x} \cos \beta x \\ e^{\alpha x} \sin \beta x \\ e^{-\alpha x} \cos \beta x \\ e^{-\alpha x} \sin \beta x \end{cases}$$
(122)

and the arbitrary constants are

$$\{C_1\} = \begin{cases} C_{11} \\ C_{12} \\ C_{13} \\ C_{14} \\ C_{15} \\ C_{16} \end{cases} : \{C_2\} = \begin{cases} C_{21} \\ C_{22} \\ C_{23} \\ C_{24} \\ C_{25} \\ C_{26} \end{cases}$$
 (123)

Only six of the twelve constants  $C_{1i}$  and  $C_{2i}$  are arbitrary; the other six are determined by substitution into Equation (112). After some lengthy algebraic manipulations, one obtains:

where  $\{C\}$  is another vector of arbitrary constants and

$$[T_1] = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & \delta_2 & -\delta_3 & 0 & 0 \\ 0 & 0 & \delta_3 & \delta_2 & 0 & 0 \\ 0 & 0 & 0 & 0 & \delta_2 & \delta_3 \\ 0 & 0 & 0 & 0 & -\delta_3 & \delta_2 \end{bmatrix}$$
(125)

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$$[T_{2}] = \begin{bmatrix} \overline{\delta}_{1} & 0 & 0 & 0 & 0 \\ 0 & \delta_{1} & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

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(125)

in which

$$\delta_{1} = \frac{p_{2}}{p_{1} + \lambda^{4}}$$

$$\delta_{2} = \frac{-(\alpha^{2} - \beta^{2} - p_{3})p_{4}}{(\alpha^{2} - \beta^{2} - p_{3})^{2} + 4\alpha^{2}\beta^{2}}$$
(126)
$$\delta_{3} = \frac{-2\alpha\beta p_{4}}{(\alpha^{2} - \beta^{2} - p_{3})^{2} + 4\alpha^{2}\beta^{2}}$$

Substituting Equations (124) into Equation (121) gives the homogeneous solution as

$$\phi = \{f\}^T [T_1] \{C\}$$

$$\theta = \{f\}^T [T_2] \{C\}$$
(127)

It will be convenient, for computation purposes, to determine the derivatives of  $\{f\}^T$  as

$$\left\{f'\right\}^{T} = \left\{f\right\}^{T} \left[\mathsf{d}\right] \tag{128}$$

where

	Γλ	0	0	0	0	0
	0	$-\lambda$	0	0	Ŋ	0
(d) -	0	0	α	β	0	0
[u] -	0	0	$-\beta$	α	0	0
	0	0	0	0	-α	β
	0	0	0	0	$-\beta$	-α

The reader will note that, for the special case of equal adherends in the single lap, one has

Special Case-Equal Adherends in a Single Lap

$$p_2 = p_4 = 0$$
$$\Gamma_1 = p_3$$
$$\lambda = \sqrt{p_3}$$

$$\alpha = \beta = \sqrt[4]{\frac{p_1}{4}}$$
$$\delta_1 = \delta_2 = \delta_3 = 0$$

and, hence, the homogeneous differential equations uncouple.

## II.S.c. General Solution

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The general solution (homogeneous plus particular) to Equations (107) will be obtained by the method of variation of parameters.<sup>(3)</sup> According to this method, one supposes that the arbitrary constants  $\{C\}$  in the homogenous solution [Equation (127)] are functions of x; i.e.,

$$\phi = \{f(x)\}^T [T_1] \{C(x)\}$$

$$\theta = \{f(x)\}^T [T_2] \{C(x)\}$$
(130)

Successive differentiations of Equations (130) are made and conditions employed on these derivatives to ensure satisfaction of the differential Equations (107):

$$\phi' = \{f'\}^{T} [T_{1}] \{C\} + \{f\}^{T} [T_{1}] \{C'\}$$
Condition 1:  $\{f\}^{T} [T_{1}] \{C'\} = q_{4}$ 

$$\theta' = \{f'\}^{T} [T_{2}] \{C\} + \{f\}^{T} [T_{2}] \{C'\}$$
Condition 2:  $\{f\}^{T} [T_{2}] \{C'\} = 0$ 

$$\phi'' = \{f''\}^{T} [T_{1}] \{C\} + \{i'\}^{T} [T_{1}] \{C'\} + q'_{4}$$
Condition 3:  $\{f'\}^{T} [T_{1}] \{C'\} = q_{3}$ 

$$\theta'' = \{f''\}^{T} [T_{2}] \{C\} + \{f'\}^{T} [T_{2}] \{C'\}$$
Condition 4:  $\{f'\}^{T} [T_{2}] \{C'\} = q_{2}$ 

$$\theta''' = \{f'''\}^{T} [T_{2}] \{C\} + \{f''\}^{T} [T_{2}] \{C'\} + q'_{2}$$
Condition 5:  $\{f''\}^{T} [T_{2}] \{C'\} = 0$ 

$$\rho'''' = \{f''''\}^{T} [T_{2}] \{C\} + \{f''''\}^{T} [T_{2}] \{C'\} + q'_{2}$$
Condition 6:  $\{f''''\}^{T} [T_{2}] \{C'\} = q_{1}$ 

In summary, the six conditions on  $\{C'\}$  are

$$[h(x)] \ \{C'(x)\} = \{q(x)\}$$
(132)

where

and the state of the

$$\left\{ q(x) \right\} = \begin{cases} q_4 \\ 0 \\ q_3 \\ q_2 \\ 0 \\ q_1 \\ \end{cases}$$
(133)

and [h(x)] is the 6 X 6 matrix

$$[h(x)] = \begin{cases} \left\{f\right\}^{T} |T_{1}| \\ \left\{f\right\}^{T} |T_{2}| \\ \left\{f'\right\}^{T} |T_{1}| \\ \left\{f'\right\}^{T} |T_{2}| \\ \left\{f''\right\}^{T} |T_{2}| \\ \left\{f'''\right\}^{T} |T_{2}| \\ \left\{f''''\right\}^{T} |T_{2}| \end{cases}$$
(134)

Solving for  $\{C(x)\}$  from Equation (132) gives

$$\{C(x)\} = \{C_0\} + \int_0^x [h(v)]^{-1} \{q(v)\} dv$$
(135)

where  $C_0$  is a vector of arbitrary constants and y is a variable of integration. (It is a consequence of this application of the method of variation of parameters that it is not necessary to evaluate derivatives of  $q_2$  and  $q_4$ .) Thus, the complete general solution is given by Equations (130) and (135):

$$\phi = \{f(x)\}^{T} [T_{1}] \left(\{C_{0}\} + \int_{0}^{x} [h(v)]^{-1} \{q(v)\} dv\right)$$

$$\theta = \{f(x)\}^{T} [T_{2}] \left(\{C_{0}\} + \int_{0}^{x} [h(v)]^{-1} \{q(v)\} dv\right)$$
(136)
(136)

The reader can satisfy himself that Equation (136) indeed satisfies Equation (107) by substitution and use of the conditions (131).

## 11.5 d. Boundary Conditions for Single, Double, and Step Lap Joints

According to the developments of Section II.2, II.3, and II.4, the boundary conditions are specified as

$$\begin{array}{ll} \underline{x} = 0 & \underline{x} = c \\ \phi = \phi_0 & \phi = \phi_c \\ \theta = \theta_0 & \theta = \theta_c \\ \theta' = \theta'_0 & \theta' = \theta'_c \end{array} \tag{137}$$

where  $\phi_0$ ,  $\theta_0$ , etc., are specified values of  $\phi$  and  $\theta$  at the boundaries. Substituting these conditions into Equations (136) and solving for the arbitrary constants  $\{C_0\}$  gives:

$$C_{0} = [H]^{-1} \left\{ \left\{ \phi_{0} \right\}^{-1} = \left\{ f(c) \right\}^{T} [T_{1}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] [T_{2}] \int_{0}^{C} [h]^{-1} \{q\} dy \\ \left\{ f'(c) \right\}^{T} [T_{2}] [T_{2}]$$

in which

 $\left\{\phi_{0}\right\} = \left\{\begin{array}{c}\phi_{0}\\\theta_{0}\\\theta_{0}\\\theta_{0}\\\phi_{c}\\\theta_{c}\\\theta_{c}\\\theta_{c}\\\theta_{c}\end{array}\right\}$ (139)

and [*H*] is the  $6 \times 6$  matrix

$$[H] = \begin{bmatrix} \{f(0)\}^{T} [T_{1}] \\ \{f(0)\}^{T} [T_{2}] \\ \{f(0)\}^{T} [T_{2}] \\ \{f(c)\}^{T} [T_{1}] \\ \{f(c)\}^{T} [T_{2}] \\ \{f'(c)\}^{T} [T_{2}] \end{bmatrix}$$
(140)

By substituting Equation (138) into (136), one obtains the particular solution of Equations (107). By defining the matrix [F] by

$$[F(x)] = [h(x)] [H]^{-1}$$
(141)

this solution can be written as

$$\begin{cases} \phi \\ \theta \end{cases} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 \end{bmatrix} [F(x)] \quad (\{\phi_0\} + \{\phi_p(x)\})$$
(142)

where

l

$$\left\{\phi_{p}(x)\right\} = \left\{\begin{array}{c} \int_{0}^{x} \mathcal{Q}_{1}(v) \, \mathrm{d}v \\ \int_{0}^{x} \mathcal{Q}_{2}(v) \, \mathrm{d}v \\ \int_{0}^{x} \mathcal{Q}_{2}(v) \, \mathrm{d}v \\ \int_{0}^{x} \mathcal{Q}_{3}(v) \, \mathrm{d}v \\ \int_{0}^{x} \mathcal{Q}_{3}(v) \, \mathrm{d}v \\ \int_{c}^{x} \mathcal{Q}_{4}(v) \, \mathrm{d}v \\ \int_{c}^{x} \mathcal{Q}_{5}(v) \, \mathrm{d}v \\ \int_{c}^{x} \mathcal{Q}_{5}(v) \, \mathrm{d}v \\ \int_{c}^{x} \mathcal{Q}_{6}(v) \, \mathrm{d}v \\ \int_{c}^{x} \mathcal{Q}_{6}(v) \, \mathrm{d}v \end{array}\right\}$$
the vector  $\left\{\mathcal{Q}(v)\right\}$ :

and the quantities  $Q_i(v)$  are elements of the vector  $\{Q(v)\}$ :

$$\{Q(y)\} = [F(y)]^{-1} \{q(y)\}$$
(144)

The integrals in  $\{\phi_p(x)\}\$  will be evaluated numerically by a standard IBM integration subroutine. The adhesive shear and normal stresses are obtained as derivatives of  $\phi$  and  $\theta$ , e.g., Equations (2) and (5). Using the conditions in Equation (131), the complete solution can be summarized as

$$\begin{pmatrix} \phi \\ 0 \\ 2\tau \\ \theta' \\ 2\sigma \\ 2\sigma' \end{pmatrix} = |F(x)] \left( \left\{ \phi_0 \right\} + \left\{ \phi_p(x) \right\} \right) + \left\{ \begin{matrix} 0 \\ 0 \\ q_4 \\ 0 \\ q_2 \\ q_2' \\ q_$$

The application of these equations to the solution of the joint problems will be discussed in Section II.5.e. II.5.d.(1) Single Lap Joint

Equation (145) applies directly to the single lap joint with the quantities  $p_i$  and  $q_i$  appropriately defined as in Section II.2.a. From Equation (73), the vector  $\{\phi_0\}$  is given by:

$$\phi_{0} = P \begin{cases} -1 \\ \frac{1}{2} & \frac{c\overline{t}}{a} - t_{1} \\ -\frac{\overline{t}}{a} \\ 1 \\ \frac{1}{2} & \frac{c\overline{t}}{a} - t_{2} \\ \frac{\overline{t}}{a} \\ \end{array} \right)$$

$$(146)$$

H.5.d.(2) Double Lap Joint

For the double lap joint, Equation (145) is used in conjunction with the boundary condition vector

$$\{\phi_0\} = P \left\{ \begin{array}{c} -1 \\ -\frac{t_1}{2} \\ 0 \\ 1 \\ \frac{t_1}{2} \\ 0 \end{array} \right\}$$
(147)

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### II.5.d.(3) Step Lap Joint

For the step lap joint, Equation (145) applies to each of the R treads. The continuity conditions [Equation (103)] which are to be applied at each intermediate riser, r, can be written by employing Equation (145) as:

$$[R_{r-1}][F_{r-1}(b_{r-1})] [\{\phi_0\}_{r-1} + \{\phi_p(b_{r-1})\}_{r-1}] = [S_r][F_r(0)] [\{\phi_0\}_r + \{\phi_p(0)\}_r]$$
(148)

where

$$[R_{r-1}] = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ \frac{h}{2} & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & \frac{hG(1-\nu^2)}{2E} \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$
$$[S_r] = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ -\frac{h}{2} & 1 & 0 & 0 & 0 & 0 \\ -\frac{h}{2} & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & \frac{-hG(1-\nu^2)}{2E} \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \end{bmatrix}$$

(149)

This provides a set of 6(R-1) equations for determining the six elements in the R vectors  $\{\phi_0\}_r$ . The additional six equations are obtained from the boundary conditions in Equation (106) which specify  $\phi$ ,  $\theta$ , and  $\theta'$  at the end risers.

### II.5.e. Iteration Procedure

The iteration procedure which will be used to solve the system of nonlinear equations was outlined briefly in Section II.5.a. A more detailed description is given here. The computer program which performs the calculations is organized along these lines. The process given on the following page refers specifically to the single lap solution. The double lap solution is almost identical. Comments regarding the step lap solution are enclosed in parenthesis.

- $(\Lambda)$  Input parameters.
  - (1) Geometry (Steps).
  - (2) Loads.
  - (3) Material constants.
  - (4) Laminate layup.

### (B) Compute appropriate constants:

- (1) Adherend stiffnesses [Eq. (60)].
- (2) Differential equation constants  $p_i$  [Eq. (71)].
- (3) Characteristic roots [Eq. (119)].
- (4) Coupling coefficients [Eq. (126)].

(This is done for each tread of the step lap.)

- (C) Subdivide the joint into a number of stations and set up the [F(x)] and  $[F(x)]^{-1}$  matrix at each station:
  - (1) Set up  $\{f(x)\}$  at each station [Eq. (122)].
  - (2) Compute the derivatives at each station [Eq. (128)].
  - (3) Set up [h(x)] at each station [Eq. (134)].
  - (4) Set up [H] [Eq. (140)] and invert.
  - (5) Obtain [F(x)] at each station [Eq. (141)].
  - (6) Invert [F(x)] at each station.

(This is done for each tread of the step lap.)

(D) Set up the boundary condition vector [Eq. (146)]:

(For the step lap, set up the continuity condition matrix at each of the intermediate risers [Eq. (148)]. Invert the coefficient matrix. Introduce the boundary conditions [Eq. (106)].)

- (E) Initialize all iteration quantities-plastic stresses, plastic stress resultants, equivalent plastic strain,  $\beta_{ij}$  coefficients, and  $\nu_p$ .
- (F) Compute the particular integrals in the differential equation solution:
  - (1) Compute the  $q_i$  at each station [Eq. (71)].

(2) Find the vector  $\{Q\}$  at each station [Eq. (144)].

(3) By numerical integration, obtain the particular integrals  $\{\phi_p\}$  [Eq. (143)]. This integration is performed by the *QSF* subroutine which is based on Simpson's rule together with Newton's 3/8 rule (see listing for details). Truncation error is of order  $n^+$ , where n is the distance between stations.

(This is done for each tread of the step lap.)

(G) Compute the adhesive stresses and adherer d stress resultants [Eqs. (145) and (12)]:

(For the step lap, insert the computed  $\{\phi_p\}$  r into the continuity conditions [Eq. (148)]. By matrix multiplication, obtain the unknown elements in the R boundary condition vectors  $\{\varphi_0\}$ . Note that the coefficient matrix was inverted in Step (D).)

(H) Find new values of the plastic stresses and stress result  $\cdot$  ts at each station:

#### Adhesive Plastic Stresses

- (1) Obtain the adhesive strains using the previous plastic stresses [Eq. (20)].
- (2) Compute the equivalent strain using the previous  $\nu_p$  [Eq. (26)].
- (3) Using the constant strain method<sup>(1)</sup>, determine the equivalent stress [Eq. (17)] and, hence, the new  $E_s$  and  $\nu_p$  [Eqs. (21) and (22)].
- (4) Determine the new plastic stresses [Eq. (28)].
- (5) Compute the iteration error as

$$e = \operatorname{Max} \frac{\left|\sigma_p^k - \sigma_p^{k-1}\right|}{\sigma} , \quad \sigma \neq 0$$

### Adherend Plastic Stress Resultants

- (1) Obtain the strain in each layer using the previous plastic stress resultants [Eqs. (32) and (33)]. For an orthotropic adherend, transform the strain to the principal material directions [Eq. (59)].
- (2) Compute the equivalent strain using the previous  $\beta_{ij}$  [Eq. (53)].
- (3) Using the constant strain method, determine the equivalent stress [Eq. (40)] and, hence, new  $\alpha_{ij}$  [Eq. (46)].
- (4) Determine the secant compliance  $S_{ijs}$  and stiffness  $Q_{ijs}$  elements [Eqs. (50) and (52)].
- (5) Compute the stresses in this layer [Eq. (51)].
- (6) Compute new values for  $\beta_{ij}$  [Eq. (54)].
- (7) Find the plastic stresses [Eq. (56)]. For orthotropic materials, transform plastic stresses to the joint plane [Eq. (57)].

(8) Compute the iteration error as

.

$$e = \operatorname{Max} \frac{\left| N_p^k - N_p^{k-1} \right|}{N_p} \quad , \quad N_p \neq 0$$

- (9) Determine the plastic stress resultants [Eq. (62)].
- (1) If the maximum error is greater than prescribed, return to Step (F) and continue iteration. Otherwise, iteration is complete and the current stresses are the final values for the prescribed load.

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## SECTION III

## DISCRETE ELEMENT ANALYSIS (DEVELOPMENT)

## III.1. GENERAL

To solve the nonlinear composite joint problem by the discrete element method, several assumptions are made:

- (1) The joint is assumed to be in a state of plane strain in the x-y plane (see Fig. 1). This assumption, or that of plane stress, is a practical necessity if any real problem is to be worked with a reasonable computer cost. A three-dimensional analysis is theoretically possible but the number of nodal points increases so rapidly that external storage devices, with a significant increase in cost, would have to be used with the digital computer. It is felt that the plane strain assumption more closely approximates the conditions along the centerline of the joint than the plane stress assumption.
- (2) The adhesive is assumed to be an isotropic material which obeys the Von Mises yield condition and the associated flow rule. The composite material is assumed to be orthotropic with transverse isotropy, i.e., isotropic in a plane perpendicular to the fibers. Superposition of plastic strains is assumed valid so that, for example, the plastic strain in the t direction resulting from a stress in the  $\ell$  direction is independent of the other stress levels.
- (3) Deformation theory of plasticity is assumed valid.

## **III.2. CONSTITUTIVE EQUATIONS**

#### III.2.a. Isotropic Material, Plane Strain

The adhesive material is considered to be an isotropic material in a state of plane strain in the x-y plane. Stresses and strains in the adhesive are represented by

where x - y is the longitudinal cross-section plane of the joint so that  $\epsilon_z = \gamma_{zy} = \gamma_{xz} = 0$  for plane strain. One can write the total strains  $\langle \epsilon_x \rangle$  as the sum of the elastic and plastic strains, i.e.,

$$\langle \epsilon_x \rangle = \langle \epsilon_x \rangle_e + \langle \epsilon_x \rangle_p \tag{151}$$

It will be noted that, in order to write (151),  $\epsilon_z$ , the strain perpendicular to the longitudinal cross-section plane of the joint, must be included. Thus,

$$\left\{\epsilon_{x}\right\}_{e}^{T} = \left\{\epsilon_{xe}\right\} \quad \epsilon_{ye} \quad \epsilon_{ze} \quad \gamma_{xze} \quad \gamma_{zye} \quad \gamma_{xye}\right\}$$
(152)

and, similarly, for  $\langle \epsilon_x \rangle \frac{T}{p}$ . Therefore, though the total strains vanish

$$\epsilon_z = \gamma_{xz} = \gamma_{zv} = 0 \tag{153}$$

for plane strain, this does not imply that the elastic and plastic strains, separately, vanish

$$\epsilon_{ze} = \gamma_{xze} = \gamma_{zye} = 0$$
  

$$\epsilon_{zp} = \gamma_{xzp} = \gamma_{zyp} = 0$$
NOT TRUE

For the isotropic material, since there is no shear coupling, it happens that the elastic and plastic shear strains are zero, although  $\epsilon_{ze}$  and  $\epsilon_{zp}$  are, in general, not zero. For the orthotropic material which has shear coupling, the elastic and plastic shear strains will not, in general, be zero.

By Hooke's law, the stresses can be found as

in which

$$[C] = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0\\ & 1-\nu & \nu & 0 & 0 & 0\\ & & 1-\nu & 0 & 0 & 0\\ & & & \frac{(1-2\nu)}{2} & 0 & 0\\ & & & \frac{(1-2\nu)}{2} & 0\\ & & & \frac{(1-2\nu)}{2} \end{bmatrix}$$

where E and  $\nu$  are the elastic modulus and Poisson's ratio, respectively. The material is assumed to obey the Von Mises yield condition. By the deformation theory of plasticity,

$$\{\epsilon_x\}_p = [S]_p \{\sigma_x\}$$
(155)

in which

$$[S]_{p} = \frac{\overline{\epsilon}_{p}}{\overline{\sigma}} \begin{bmatrix} 1 & -\frac{1}{2} & -\frac{1}{2} & 0 & 0 & 0 \\ 1 & -\frac{1}{2} & 0 & 0 & 0 \\ & 1 & 0 & 0 & 0 \\ & & 1 & 0 & 0 & 0 \\ & & & 3 & 0 & 0 \\ Symmetric & & & 3 & 0 \\ & & & & & 3 \end{bmatrix}$$

where

$$\overline{\sigma} = \begin{cases} \frac{1}{2} \left[ (\sigma_x - \sigma_z)^2 + (\sigma_z - \sigma_y)^2 + (\sigma_y - \sigma_x)^2 \right] + 3 \left[ \tau_{xz}^2 + \tau_{zy}^2 + \tau_{xy}^2 \right] \begin{cases} 1/2 \\ 1/2 \end{cases}$$

is the effective stress and  $\overline{\epsilon_p}$  is the effective plastic strain which may be found from the material stress-strain curve in uniaxial stress. If the Ramberg-Osgood law<sup>(2)</sup> is used to approximate the stress-strain curve, one has

$$\bar{\epsilon} = \frac{\bar{\sigma}}{E} + \frac{3\sigma_0}{7E} \left(\frac{\bar{\sigma}}{\sigma_0}\right)^n \tag{156}$$

in which

 $\sigma_0$  = secant yield stress (stress at which the secant modulus = 0.7*E*)

$$n = \text{shape factor} = 1 + \frac{\log\left(\frac{17}{7}\right)}{\log\left(\frac{\sigma_0}{\sigma_1}\right)}$$

where

 $\sigma_1$  = stress at secant modulus at 0.85*E* 

According to the constant strain method<sup>(2)</sup>, the effective plastic strain  $\overline{\epsilon}_p$  for a given total effective strain  $\overline{\epsilon}$  can be found from Equation (156) if it is rewritten as

$$\bar{c} = \left(\frac{7\bar{\epsilon}_p}{3}\right)^{1/n} \left(\frac{\sigma_0}{E}\right)^{1-\frac{1}{n}} + \bar{\epsilon}_p \tag{157}$$

The above expressions could be simplified slightly by eliminating the elastic and plastic shearing strains  $\gamma_{xz}$  and  $\gamma_{zy}$  since this is an isotropic material. However, in order to keep the development parallel to the orthotropic case, they are included.

## III.2.b. Orthotropic Material, Plane Strain

One ply (or lamina) of the adherend composite material, Figure 6, is assumed to be orthotropic. The principal material directions are  $\ell$ -*t*, *v* where  $\ell$  is parallel to the fiber direction. Stress and strains in the principal material directions are

The composite is assumed to be isotropic in the t-y plane.

Separating the total strain into its elastic and plastic components, one can write

$$\left\{\epsilon_{\varrho}\right\} = \left\{\epsilon_{\varrho}\right\}_{e} + \left\{\epsilon_{\varrho}\right\}_{p} \tag{159}$$

Using Hooke's law, one obtains the stresses as

$$\{\sigma_{\varrho}\} = [C] \{\epsilon_{\varrho}\}_{\varrho} \tag{160}$$



FIGURE 6. ORTHOTROPIC LAMINA COORDINATES *ℓ*, *t*, *y* AND STRESS-STRAIN CURVES FROM UNI-AXIAL TEST IN PRINCIPAL DIRECTIONS where



All the  $E_i$  values are lamina elastic constants (some negative) which are obtained as the initial slope of the *i*th curve in Figure 6.

According to assumption (2) in Section 111.1, it is assumed that the plastic strains can be found by superposition, e.g.:

$$\epsilon_{\ell p} = \epsilon_{\ell \ell p} + \epsilon_{\ell p} + \epsilon_{\ell y p}$$

where  $\epsilon_{\ell p}$  is the longitudinal plastic strain corresponding to the stress  $\sigma_t$  as found from the simple uniaxial test. Let the stress-strain relations from uniaxial tests of a typical lamina be as given in Figure 6. Each of the seven curves car be approximated by a Ramberg-Osgood relation of the form

$$\epsilon = \frac{\sigma}{E_i} + \frac{3\sigma_{0i}}{7E_i} \left(\frac{\sigma}{\sigma_{0i}}\right)^{n_i} \tag{161}$$

where  $\epsilon$  and  $\sigma$  are the appropriate stress and strain for each curve,  $E_i$  is the initial stope of the curve,  $\sigma_{0i}$  and  $n_i$  are the quantities corresponding to those given in Equation (156) and *i* refers to the curve number as given in Figure 6. The plastic strains can now be written as\*

$$\begin{cases} \epsilon_{vp} \\ \epsilon_{ip} \\ \epsilon_{vp} \\ \epsilon_{vp} \\ r_{vp} \end{cases} = \frac{3}{7} \begin{cases} \frac{\sigma_{01}}{E_1} \left(\frac{\sigma_{01}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{01}}{E_1} \left(\frac{\sigma_{1}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{01}}{E_1} \left(\frac{\sigma_{1}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{01}}{E_1} \left(\frac{\sigma_{1}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{01}}{E_1} \left(\frac{\sigma_{1}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{02}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{02}}\right)^{n_1} + \frac{\sigma_{03}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{01}}\right)^{n_1} + \frac{\sigma_{03}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{02}}\right)^{n_1} + \frac{\sigma_{04}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{02}}\right)^{n_1} + \frac{\sigma_{04}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{02}}\right)^{n_1} + \frac{\sigma_{04}}{E_2} \left(\frac{\sigma_{2}}{\sigma_{02}}\right)^{n_1} \end{cases}$$

$$\end{cases}$$

$$(162)$$

$$\begin{cases} \sigma_{02} \left(\frac{\sigma_{1}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{02}}\right)^{n_1} + \frac{\sigma_{05}}{E_2} \left(\frac{\sigma_{1}}{\sigma_{03}}\right)^{n_1} + \frac{\sigma_{04}}{E_4} \left(\frac{\sigma_{2}}{\sigma_{04}}\right)^{n_1} + \frac{\sigma_{2}}{E_4} \left(\frac{\sigma_{2}}{\sigma_$$

\*The plasticity theory used here for orthotropic materials is different than that employed in the theoretical methods. The "best" theory has not yet been established by experiments. As the results from the two theories show (see the following section), the difference is insignificant.

The transformation relating the plane strain strains in the x-y plane to the strains in the principal material directions  $\nabla t$ -y can be written as

$$\langle \epsilon_{\varrho} \rangle = [T] \quad \langle \epsilon_{\chi} \rangle \tag{163}$$

where  $\langle e_x \rangle$  and  $\langle e_x \rangle$  are given in Equations (158) and (150), respectively, and [T] is a 6 × 3 matrix:

	$\cos^2\psi$	0	0 ]
	$\sin^2\psi$	0	0
[T] -	0	I	0
[1] -	$-2 \sin \psi \cos \psi$	0	0
	0	0	$-\sin\psi$
	0	0	cosψ

in which  $\psi$  is the angle of fiber orientation as shown in Figure 6.

## 111.3. ELEMENT STIFFNESS MATRIX AND PLASTIC FORCES

After the constitutive equations have been defined in the above manner, the solution of the discrete element problem follows very closely the procedure outlined for ELPLAN<sup>(4)</sup>, a computer program for the inelastic analysis of plane stress problems. A typical finite element is taken to be a triangle in the longitudinal cross-section plane of the joint (x-y plane) with a unit thickness, Figure 7. The following expressions are obtained for the element ctiffness



ELEMENT USED FOR NONLINEAR JOINT ANALYSIS matrix [k] and the plastic nodal forces  $\{f_p\}$  for the composite material:

$$[k] = \overline{A}[B]^{T}[T]^{T}[C][T][B]$$
(164)

$$\{f_p\} = [D] \quad \{\epsilon_{\mathcal{Q}}\}_p \tag{165}$$

$$[D] = \overline{A}[B]^T[T]^T[C] \tag{166}$$

in which [T] is given in Equation (163), [C] in Equation (160),  $\langle \epsilon_{\varrho} \rangle_p$  in Equation (162) and  $\overline{A}$  is the element area in the x-y plane. The matrix [B] relates the element strains to the nodal displacements  $\langle X \rangle^{(4)}$ , i.e.,

$$\{\epsilon_X\} = [B] \quad \{X\} \tag{167}$$

The stresses in a composite element in the principal material directions are

$$\left\langle \sigma_{\varrho} \right\rangle = [C] \left( [T] [B] \left\{ X \right\} - \left\{ \epsilon_{\varrho} \right\}_{p} \right) \tag{168}$$

The above expressions apply also to the isotropic adhesive material although they can be slightly simplified. For programming purposes, it is convenient to use the same algorithm for the isotropic and orthotropic materials. In this regard, Equations (164), (165), (166), (167), and (168) are valid for the

adhesive if  $\psi$  is taken equal to zero in [T]. Then [C] is found in Equation (154), and Equation (155) is used for the plastic strains.

The solution of the discrete element problem proceeds, according to deformation theory, in the following manner:

- (1) Formulate the structural stiffness matrix [K] by assembling the element stiffness matrices in (164) and the applied nodal load matrix  $\{F\}$  as specified by the loading.
- (2) Perform an elastic analysis with the current values of  $\{F_p\}$  to obtain the nodal displacements  $\{X\}$ :

$$\langle F \rangle + \langle F_p \rangle = [K] \langle X \rangle$$

For the initial iteration,  $\langle F_p \rangle = \langle O \rangle$ .

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- (3) Calculate the stresses via Equation (168) with the current values of  $\langle \epsilon_{\varrho} \rangle_p$ ,  $\langle \epsilon_{\varrho} \rangle_p = 0$  for the first iteration.
- (4) Calculate the new plastic strains via Equation (162) for the composite or (155), (156), and (157) for the adhesive.
- (5) Consider the new plastic strains as initial strains and compute new values of the element plastic forces  $\{f_p\}$  by Equation (165). Assemble the element plastic forces into the structural plastic forces  $\{F_p\}$ .
- (6) If the maximum change in plastic strain from the previous iteration is sufficiently small, the solution is complete. If not, return to Step 2.

This process was incorporated into ELPLAN. The application of the resulting program is discussed in a later section.

## SECTION IV

## COMPARISON OF THEORETICAL AND DISCRETE ELEMENT RESULTS

### IV.1. JOINT CONFIGURATIONS AND MATERIAL PROPERTIES

In order to compare the theoretical and discrete element analysis methods, the three particular joint configurations shown in Figure 8, i.e., single lap, double lap, and step lap, were analyzed by both methods. The Narmco 5505 Boron/Epoxy System was taken as the adherend and AF-126-2 as the adhesive. The adherend was a



### MATERIAL CONSTANTS FOR RAMBERG-OSGOOD APPROXIMATION

Adhesive (AF-126-2)								
	G (ksi)	$ au_0$ (ksi)	п					
τνς γ	175	3.32	2.684					
Adherend (Narmco 5505)								
Curve (i) of Figure 6	$E_i$ (ksi)	σ <sub>01</sub> (ksi)	n <sub>i</sub>					
l σ <sub>ℓ</sub> vs ε <sub>ℓ</sub>	29,600	312.7	4.463					
2 $\sigma_{\varrho} vs \epsilon_t (\epsilon_y)$	-130,000	285.5	5.129					
3 $\sigma_t(\sigma_y)$ vs $\epsilon_{\ell}$	-1 30,000	285.5	5.129					
4 $\sigma_t(\sigma_y)$ vs $\epsilon_t(\epsilon_y)$	2,750	1.91	2.541					
5 $\sigma_t(\sigma_y)$ vs $\epsilon_y(\epsilon_t)$	-8,876	10.52	3,350					
6 $\tau_{\ell t}(\tau_{\ell y})$ vs $\gamma_{\ell t}(\gamma_{\ell y})$	933	7.95	2.991					
7 $\tau_{ty}$ vs $\gamma_{ty}$	191	68.44	2.031					



6.25

COMPARISON OF ANALYSIS METHODS

five-ply laminate (nominal ply thickness 0.0052 in.) with  $0^{\circ}$ ,  $90^{\circ}$ ,  $0^{\circ}$ ,  $0^{\circ}$ ,  $0^{\circ}$  orientations, except for the inner laminate of the double lap which had nine plies with 0/90 orientations. In each case, the joint length, c, was 0.75 in., and the total length, a, was 6.25 inches. The adhesive thickness was 0.005 inch.

The material properties used in the Ramberg-Osgood approximation [Eq. (156)] of the adhesive *shear* stressstrain curve are shown in Table I. Poisson's ratio of the adhesive was taken as 0.3. The material constants for the characterization of a typical lamina of an adherend are also given in Table I. They represent the Ramberg-Osgood constants in Equation (161) for the uniaxial stress-strain curves in Figure 6. The stress-strain curve for AF-126-2 was not a vailable at the time of the comparison, but those values shown in Table I were thought to be appropriate.<sup>(5)</sup> More recent work shows that the shear modulus is about 80 ksi<sup>(6)</sup> instead of the 175 ksi value shown in Table I and used in the analysis. Since consistent material properties were used in both analysis methods, the results of the comparison study will remain valid, however. Stress-strain curves for the Narmoo 5505 were obtained from Reference 7. Curves 5 and 7 for lamina characterization in a plane perpendicular to the fibers were not available. These curves were assumed to be identical to those of the transverse unidirectional lamina. Many of the curves for the lamina were quite linear to failure, and, thus, do not reach the stress  $\sigma_0$  which corresponds to a secant modulus of 0.7E. In these cases, the value of  $\sigma_0$  and *n* were determined such that the Ramberg-Osgood approximation passed through two points on the upper nonlinear portion of the stress-strain curves. For an isotropic material, the solution for n and  $o_0$  is

$$n = \left(\frac{E\epsilon' - \sigma'}{E\epsilon'' - \sigma''}\right) / \left(\frac{\sigma'}{\sigma''}\right)$$
$$\sigma_0 = \left[\frac{\gamma}{3} \left(E\epsilon'' - \sigma''\right)(\sigma')^{-n}\right]^{\frac{1}{1-n}}$$

where  $(\sigma', \epsilon')$  and  $(\sigma'', \epsilon'')$  are two points on nonlinear portion of the stress-strain curve.

## IV.2. JOINT ANALYSIS

### IV.2.a. Discrete Element Analysis

The three joints shown in Figure 8 were analyzed by the discrete element computer program discussed in Section III. The finite element idealization of the single, double, and step lap joint are shown in Figures 9, 10, and 11, respectively. Joint boundary conditions are illustrated schematically in these figures. Material properties listed in Table I were used.

Results from the discrete element analysis for the shear and normal stress in the adhesive are presented as circled points in Figures 12, 13, and 14 for the single, double, and step lap joints, respectively. The discrete element program evaluates the stresses at the centroid of each triangular element. Hence, the stresses are evaluated at 1/3 and 2/3 thickness levels in the adhesive. In order to compare results with the theoretical method, these stresses were averaged to obtain the stress at midthickness. The discrete element program did not converge to the specified error tolerance of 0.01 within 20 iterations for step lap joint loads greater than 1000 lb/inch. This was caused by large plastic laminate strains developing in the stress concentration area at the juncture of a tread and a riser. (Note the change in the stress scale for the step lap joint as compared to that of the single and double lap joints.)

#### IV.2.b. Theoretical Analysis

The theoretical analysis technique outlined in Section II and programmed for the CDC 6400 computer was also used to analyze the joints in Figure 8. In addition to the geometric quantities, the adhesive material constants and the adherend material constants for Curves 1, 2, 4, and 6 from Table I were input into the program. These four curves for the adherend correspond to the four curves in Figure 3 and the four Equations (41a, b, c, and d). For the numerical integration involved in this solution, the single lap and double lap were subdivided into 20 equal segments (21 stations) along the joint. Each tread of the step lap was subdivided into 30 equal segments. The results of the theoretical analysis are presented as the curves in Figures 12, 13, and 14 for the three joints.

### IV.2.e. Discussion

Despite the different assumptions involved in the finite element method and the analytical method, i.e., plastic strain superpositon for the finite element method versus deformation theory for the analytical method and three dimensional stresses in the finite element method versus negligible shear deformation in the adherends for the analytical method, the comparison of the results in Figures 12 and 13 is quite good for the adhesive shear stress in the single and double lap joint. The difference in the two methods for the step lap joint adhesive shear (Fig. 14) is probably due to both of the following two causes:

- Shear deformation is neglected in the adherends for the theoretical method. For the step, the adhesive is attached to the 90-degree oriented layers. The shear modulus of these layers in the plane of the joint is only 191 ksi (see Table I), which is about equal to that of the adhesive itself.
- Transmission of force through the step risers is neglected in the theoretical method. Hence, the total force is transmitted by shear along the treads. Thus, the average shear stress for the theoretical method is about 1000/0.75 or 1333 psi, whereas it is lower for the discrete element method since some force is transmitted through the risers.



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# FIGURE 9. DISCRETE ELEMENT LAYOUT FOR SINGLE LAP JOINT



## FIGURE 10. DISCRETE ELEMENT LAYOUT FOR DOUBLE LAP JOINT













FIGURE 13. ADHESIVE STRESS, DOUBLE LAP JOINT





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According to Equation (122), the normal stress varies approximately as a damped cosine (and sine) curve with a period of length

$$L = \frac{2\pi}{\omega} \tag{169}$$

where, approximately,

$$\omega = \left[\frac{E}{4t(1-\nu^2)} \left(\frac{1}{D_1} + \frac{1}{D_2}\right)\right]^{1/4}$$
(170)

These conditions are exact for  $\pm$  e special case of equal adherends in the single lap-see Section II.2. For the single, double, and step lap joints, the c/clic period, L, of the normal stress is 0.189 in., 0.224 in., and 0.084 in., respectively. (Note that  $1/D_2 = 0$  for the double lap.) The discrete elements in the region of the adhesive were 0.025 in. long. These were not sufficiently small to pick up the rapid variation of the normal stress since the method is based on an assumed constant stress in each element. The theoretical method is somewhat limited in this regard also because of the numerical integration involved. However, the integration is performed according to Simpson's rule which is based on a parabolic approximation to the integrand. The error for the discrete element method is about  $h^2$  and about  $h^5$  for the theoretical method where h is the interval spacing. Thus, the normal stresses obtained by the theoretical method are probably more reliable than those from the discrete element method.

Although extensive calculations were not conducted for verification, it is thought that the interval for the numerical integration in the theoretical method should be no greater than about  $L/2\pi$  or  $1/\omega$ . (For the examples above, the intervals were L/5.0, L/6.0, and L/6.8 for the single, double, and step lap, respectively.

#### SECTION V

### EXPERIMENTAL DESIGN

### V.1. GENERAL

This section presents the study of bonded joint data taken from the literature followed by the design of the experimental program to verify the theory of Sections II, III, and IV. Section V.2 presents the Literature Study covering six information data sources to establish statistical techniques for experimental data analysis and provide experimental design information. Section V.3 presents Trial Effective Properties Prediction for the elastic case utilizing the methods developed in the previous three sections. Design of the Experimental Program presented in Section V.4 provides the detailed history of the specimen selection and design which were used for verification of the new analytical methods in subsequent sections.

### V.2. LITERATURE SURVEY

An example development of single and double lap joint experimental data analysis is presented in this section. It was accomplished by the study and analysis of test data from three sources: Douglas, IITRI, and SwRI. Preliminary analysis of the more limited data from two additional sources, Martin-Orlando and Grumman is also included. Two composite materials, boron/epoxy and S-glass/epoxy of several orientations were utilized as the primary adherend materials in the data studied in detail and they were bonded to the same materials or to aluminum, titanium or woven E-glass/epoxy secondary adherends. For the two sources of data studied in preliminary fashion, Grumman used composite and metal adherends while Martin-Orlando used only metal.

The data collection and analysis effort had two primary objectives: a gross characterization of the "effective" properties of the various adhesives, and a meaningful estimate of the general test precision in order to establish guidelines for the experimental effort. Pertinent data from all five sources [i.e., Dastin of Grumman<sup>(8)</sup>, Lehman of McDonnell-Douglas<sup>(9)</sup>, Chessin and Curran of Martin-Orlando<sup>(10)</sup>, Kutscha of IITRI<sup>(11)</sup>, and Grimes of SwRi<sup>(12)</sup>] yielded lap joint test data in quantities sufficient for meaningful analysis. The Martin-Orlando data on metal adherend-bonded lap joints were included for comparative purposes. It should be noted that the Martin-Orlando data covered only one type of variable in their tests, the adherend surface preparation. While this variation caused a larger range of failure stress magnitudes than would be expected with one chosen surface preparation, the standard deviation was comparable to that of the Douglas, HTRI, and SwRI data which had variable overlap lengths and the Grumman data which used several different adhesives.

The data analysis consisted of two sequential steps: the generation of reliable precision estimates in order that confidence intervals might be established for the various mean failure stress measurements, and the subsequent use of these intervals to establish a reasonable range of apparent material properties as a function of the joint configuration for experimental design and data analysis purposes. The analysis work concentrated on the average apparent adhesive shear stress failure measurement, the average adherend tensile failure stresses, and on the running loads transferred in lb/in./ply. The generation of precision estimates was complicated generally by the scarcity of such estimates in the literature and specifically by the wide variance in test parameters among the five major data sources. As an example, it can be seen that, for the various parameters by which the data were tabulated, not a single instance can be found in which two different sources ran an identical test. This, of course, means that interlaboratory reproducibility could not be estimated.

On the other hand, the data collected did prove sufficient for estimating the intralaboratory repeatability, provided that analysis was approached via a method now under study for publication by ASTM Committee D2. This procedure is designed specifically for the generation of precision estimates from data in which

- (1) the standard deviation appears to vary with the mean rating of the various samples, and
- (2) only relatively small amounts of data are available on any one sample.

This is precisely the case with the lap joint data collected. The first step in the process was to calculate the standard deviations for the individual samples. Following this, a decision had to be made concerning grouping of the data, e.g., of the various parameters in the tabulation; how many should one group together? In this case, the decision was tairly easy, since groups broken down any further than source and test type would be too small to be useful. Hence, the data were divided into these six groups:

Single LapDouglas -(adherend materials and overlap length variations)Grumman--(adherend material and adhesive type variations)Martin-Orlando--(adherend material surface preparation variation)

Double LapDouglas-(adherend material and overlap length variation)SwRI-(adherend material and overlap length variation)IITRI-(adherend material, adhesive, and overlap length variation)

Linear regression lines resulting from plots of standard deviation vs mean adhesive shear stress failure for each of these six groups\* are presented in Figures 15 and 16. These straight lines were fitted to the data points by linear regression techniques which represent the best estimates of the overall test (or population) standard deviations for each of the six groups. The advantage of the above approach is that a large number of degrees of freedom can be utilized in the estimating procedure rather than the small number available in the few data points actually falling at a given level. This gives a much more realistic calculation for confidence intervals and provides a means of looking at data trends with respect to the variables encountered.

The difference between Figures 15 and 16 is the inclusion and exclusion, respectively, of the Metalbond 400 data in those analyzed from HTRI. When the Metalbond 400 data is removed from the balance of the HTRI data, the standard deviation vs mean line falls on top of all the rest of the data except for the Douglas single lap data. These two groups of data were obviously out of control in some fashion. It could have been the material, processing or testing; however, the important point is that the statistical technique picked it up.

Before dealing with confidence intervals, a study of the trends is shown in Figure 16 with the standard deviation estimates for all six data groups given on the same plot. Here it can be seen that, at mean adhesive shear stress failure levels in the area of 3,000 psi, the standard deviations for all but one are approximately equal. Considering that the experimental parameters varied a great deal in these tests, and that the double lap data behaved (statistically) much like two of the three single lap groups the general variance to be expected for both groups appears to be about the same at any mean level, regardless of the adhesive type, adherend combination, overlap length, etc.

Calculation of the confidence intervals for the population mean adhesive shear stress failure levels consists of using the regression line standard deviation estimate in the following formula:

95% Confidence Limits at 
$$f_s = f_s^{-1} \pm \frac{ts}{\sqrt{n}}$$
 (171)

where  $f_s$  is the average of these experimentally determined mean adhesive failure stresses, *n* is the number of determinations, *s* is standard deviation, and *t* is the *t*-deviate corresponding to the number of degrees of freedom involved in the *regression line estimate* of the standard deviation (not *n*). These limits define the interval within which the mean of a very large number of tests would probably lie relative to the mean of this experimental data.

With the confidence limits established, it is possible to calculate 95% confidence design allowables in the following manner:

$$DA_{95} = LCL - ts\left(\frac{1}{\sqrt{n}} + 1\right) \tag{172}$$

\*Using ASTM E-178 (Ref 13) for the deletion of "outlier points."







FIGURE 15. STANDARD DEVIATION VS MEAN ADHESIVE FAILURE STRESS-COMPARISON

where  $LCL = f_s - (ts/\sqrt{n})$  is the lower confidence limit and t, n and s have the same definitions as above. In essence, this calculation says that, if the population mean  $f_s$  did turn out to be at the lower confidence level, then about 5 of 100 specimens would fail at the design allowable stress level or lower. This is a conservative estimate; the real failure probabilities should be more favorable.

In the Douglas, IITRI and SwRI data groups, overlap lengths were varied, hence, it was possible to plot mean failure stress (or unit load) vs overlap lengths for various composite materials and joint types. These mean experimental plots are presented in Figures 17 through 20

The data analyses performed on the unit running loads per ply transferred were essentially a reiteration of those described above for adhesive failure stresses. Linear regression lines based on plots of standard deviation vs mean running load transferred in lb/in./ply for four of the five groups mentioned (the Martin-Orlando tests being on metal adherends were omitted) are presented in Figure 29. Because the adhesive failure stress and the running load are both calculated at the failure point and thus have a proportional relationship, it should be noted that their linear regressions look much alike. Only the Douglas single lap data regressions showed any appreciable difference in slope, and this is most probably due to the following related factors:

- (1) These data generally showed a significantly higher variance than did the other groups
- (2) The accuracy of the regression line slope as an estimator of the corresponding population statistic is inversely proportional to the average magnitude of the sample variance involved.

Thus, the different slope indicated for the Douglas single lap data may well be apparent rather than real, although Figure 29 shows that the standard deviations for all but the Douglas single lap data are approximately equal at a load-ing of 500 lb/in./ply.

Several interesting phenomena can be seen from study of this data. The lower curves in Figure 25 showed that a weak interface region is detrimental to composite bonded joints. The cause of this could be high adhesive viscosity at flow temperature (occurring during cure), i.e., a material quality problem, probably aging. Also, it can be seen from Figures 30 through 33 that a comparison of the plots of mean vs standard deviations for shear stress and load/ ply transferred are very similar (as would be expected) except for the Douglas single lap data. In Figure 34 the coefficient of variation line slope and location are considerably different from that of the standard deviation. Because of this the coefficient of variation is less desirable as a design tool than is the standard deviation. Finally, a plot of the 95% confidence limits of the failure stresses for the Martin-Onando data showing controlled variations in processing are significant is presented in Figure 35. This is because all the data are shown to be consistent (under control) even though the difference in the magnitude of the mean stresses is quite large as a result of different processing techniques. It illustrates that the right processing technique should be chosen and kept under close control.

Illustrations of plots of 95% confidence limits vs mean failure magnitudes are shown in Figures 36 through 40. In Figure 36 the shear stresses for the IITRI data are presented in this fashion while Figures 37 through 40 present the SwRI, Douglas, and Grumman data in terms of lb/in./ply of running load transferred.

From the study of these data from the literature, techniques have been established to analyze similar experimental information on bonded joints for acceptable scatter limits (confidence limits) for the 95% confidence level when the number of like test points are limited.

## V.3. TRIAL EFFECTIVE PROPERTIES PREDICTION<sup>(14)</sup>

Utilizing the mean data from Figure 23 on a 1/2-inch overlap double lap joint with a 14-ply  $[0]_c$  adherend made of Scotchply XP-251S the shear stress distribution was calculated for three assumed bondline shear moduli values. The *G* values for which  $\tau_x$  distribution will be computed are 160, 90, and 40 ksi.



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FIGURE 27. NGE/NGE IITRI DOUBLE LAP/FM-1000

Overlap Length, in.



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Overlap Length, in.

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FIGURE 32. COMPARISON OF STANDARD DEVIATIONS-POUGLAS DOUBLE LAP DA1A


FIGURE 34. COEFFICIENT OF VARIATION AND STANDARD DEVIATION VS MEAN FAILURE STRESS, SWRI DOUBLE LAP DATA

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FIGURE 36. CONFIDENCE LIMITS-IITRI DOUBLE LAP DATA



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FIGURE 37. CONFIDENCE LIMITS-SwRI DOUBLE LAP DATA



FIGURE 38. CONFIDENCE LIMITS-DOUGLAS SINGLE LAP DATA



FIGURE 39. CONFIDENCE LIMITS - DOUGLAS DOUBLE LAP DATA



FIGURE 40. CONFIDENCE LIMITS-GRUMMAN SINGLE LAP DATA

Figure 41 shows the distribution for the G = 160-ksi value. It is assumed that the end point  $\tau_{max}$  and  $\tau_c$  values are found at one bondline thickness (0.00315 in.) away from the end of the joint (via St. Venant's principle), for the shear stress must go to zero at both ends at the free surface. However, the  $\tau_{max}$  value of 21,600 psi shown is, in the authors' judgment, too high. If the elastic G of the adhesive is actually close to the assumed 160 ksi, the end points would have to be cut off at some lower  $\tau_{max}$  value which would make the area under the  $\tau_{x}$  curve the same as the area under the  $\tau_{avg}$  line (i.e.,  $A_1 + A_3 = A_2$  on Fig. 41).

Figure 42 looks to be a more reasonable assumption for an assumed G = 90,000 psi with theoretical  $\tau_{max}$  being closer to the estimated  $\tau_{max}$ . It should be noted that lowering the modulus lowers the maximum shear stresses in the cement and raises the minimum stresses.

One additional computation for a G = 40,000 psi is shown in Figure 43. This is made to establish the relationship between adhesive shear stresses and the assumed G for the specific composite adherend combination, joint geometry, and experimentally measured average failing shear stress utilized. When the results from Figures 41, 42, and 43 are plotted, the variation of  $\tau$  with the assumed G is shown in Figure 44. This shows the "effective" G to be 12,500 psi for  $\tau_{max} = 8,400$  psi and  $\tau_{avg} = 5,250$  psi, assuming elastic conditions to failure.

From this initial preliminary study the technique of using an effective G to determine bondline shear stress distribution was deemed feasible for the elastic condition.

# V.4. DESIGN OF THE EXPERIMENTAL PROGRAM

An experimental program designed to verify the analytical techniques developed herein requires the complete evaluation of the mechanical properties of (1) the adhesives, (2) the adherend materials, and (3) joints made from these materials. Since the first item was being evaluated by at least two other programs, effort in this contract was concentrated on Items 2 and 3.

Because of the wealth of data available on N-5505 boron/epoxy composites and 6A1-4V titanium it was decided to evaluate only the longitudinal properties of these adherend materials. Three study areas on joints were decided upon to satisfy the verification objective and those of the contract Statement of Work. These were (1) a large number of simple specimen bonded joints tested primarily to determine ultimate strength, (2) a small number of special bonded joints to evaluate the strain distribution under a monotonically increasing load to failure, and (3) a very small number of complex (larger) joints to evaluate size effects on both ultimate strength and strain distribution.

## V.4.a. Adherend Materials

It was decided to test four longitudinal tensile specimens from the two 0.938 in. wide, 20 in. long strips taken from each composite adherend panel made. In addition, two similar configuration tensile specimens were to be taken from each of the four gages of 6AI-4V titanium sheet. Specimen and test details are given in specification SwR1 03-401, Test Standard for Fibrous Composite Tensile Specimens published in Appendix C of this report. Complete uniaxial tension stress/biaxial strain curves to failure were to be recorded on each static test specimen. These data were then to be used in the nonlinear methods developed in this program for bonded joint analysis.

For purposes of experimental design, properties based on average test data were taken from the literature as follows:

	N-5505 (50%	Boron/Epo F.V. Fractio	xy <sup>(15)</sup> on)*	Sheet <sup>(16)</sup>
Property	0	0/90	0/±45	Titanium (6Al-4V) Ann.
$\sigma_{\mathcal{R}_{\mathcal{U}}}$	191.0 ksi	72.0 ksi	103.0 ksi	147.0 ksi
$\sigma_{\varrho p \varrho}$	134.0 ksi	29.0 ksi	36.7 ksi	120.0 ksi

\*0.0052 in./ply







FIGURE 42. ORTHO-/ISO-ELASTIC SHEAR STRESS DISTRIBUTION FOR G = 90,000 PSI

S-Glass/Epoxy Adherend,  $[0_{14}]_T$ ,  $t_0 = t_1$ Nitrile-Epoxy Adhesive, t = 0.00315 in. Overlap 0.50 in. P/2 to - 0,00315 in. t 12 -12/2 max = 11,500 psi (Theory) **6**0 - x 0,00315 in 7 max = 8,400 psi (Estimated) 8 <sup>τ</sup>C - 6,800 (Theory) **ks**1 τ<sub>avg</sub> = 5,250 psi ċ 4 Tx Curve 7min = 3,300 psi (Theory) 0 0.40 0.45 0,50 0.35 0.15 0.30 0.05 0,10 0.20 0.25 Overlap Length, in. X C 0 **x** =

Double Lap Joint





FIGURE 44. VARIATION OF SHEAR STRESS WITH ASSUMED SHEAR MODULUS

# V 4 b Adhesive Properties

Since little data were available on the emperical strength of AF-126-2 and MB-329 adhesives used in single, double, step lap and scarf joints with variable overlap lengths, the data which were available were used to develop extrapolated curves. Those for AF-126-2 nitrile epoxy, low stiffness-high elongation (LSHE) adhesive are shown in Figure 45. Figure 46 presents similar type curves on MB-329 epoxy novolak high stiffness-low elongation (HSLE) adhesive. These estimated ultimate shear strength values were used in the design of the joint specimens.

# V.4.c. Simple and Special Joint Specimen Design

Using the properties presented in the previous sub-sections, joint design curves were calculated and plotted which would cover both the linear and nonlinear ranges of both the adherend and the adhesive. In other words, some joints were designed to fail in the adhesive with the adherend tensile stresses in either the linear or nonlinear tange but below failure. Others were designed to fail in the adherend while the adhesive shear stress was either in the linear or nonlinear range. Additionally, some were designed to cause failure to occur simultaneously in the adhesive and the adherend.

For the three fiber orientations selected and the four types (3 lap and 1 scarf) of joints to be studied, the empirical design curves using the AF-126-2 (LSHE) adhesive are given in Figures 47 through 49. Similar curves are presented for the MB-329 (HSLE) adhesive in Figures 50 through 52. With these empirical design curves based on average test properties, the overlap lengths were designed for a given stress level in the adherend for a given type adhesive and orientation of the composite. This information was then used to generate the required number of plies of a balanced symmetric laminate. Where titanium was used as the other adherend materials it was matched as closely as possible to the total composite adherend thickness. With this information, the simple and special specimen test plan could be completed and is shown in Table II

# V.4.d. Complex Joint Design

General requirements established for the complex joints (C.J.) were:

- (1) One (1) C.J. test panel for each adhesive, i.e., two (2) C.J. test panels, total
- (2) Each C.J. test panel: approximately 5 in. wide X 15 in. long
- (3) Each C.J. test panel: double lap type
- (4) Each C.J. test panel: instrumented to determine load/strain distribution, concentrations, failure initiation locations, and ultimates.











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# TABLE II

# SIMPLE AND "SPECIAL" SPECIMEN TEST PLAN (Monotonically Increasing Load to Failure)

	General						Overla	p Segm	ents*					Line			
Type of	Fiber		Num	iber	1		Nu	mber 2			Num	bei	- 3		Total		
Joine	Drientation)		B-B		B-T		B-B	B	Т		B-B		B-T	]	rota		
	Category	Ν	0.L.	Ν	0.L.	N	0.L.	N†	0.L.	Ν	0.L.	N	0.L.		N		
Single Lap	$[0]_{c}$ $[0/90]_{c}$ $[0/\pm 45]_{c}$	3	1/4 1/4	3	1/4	.,	1-1/4 1-1/4	3 3 + 1	1-1/4	3	1-3/4 2	3 - 3	1-3/4 2	18 0 18 + 1			
Double Lap	[0] <sub>c</sub> [0/90] <sub>c</sub> [0/±45] <sub>c</sub>	3 3 3	1/4 1/4 1/4	3 3 3	1/4 1/4 1/4	3 3 3	1/2 1/2 3/4	3 3 3+1	1/2 1/2 3/4	3 3 3	3/4 3/4 1-1/4	3 3 3	3/4 3/4 1-1/4	18 18 18 † 1	AF-126-2(LS-HE) adhesive		
Step Lap	[0] <sub>c</sub> [0/90] <sub>c</sub> [0/±45] <sub>c</sub>	1 11	- W	3 3 3	1/4 1/4 1/4		-	$3 \\ 3 \\ 3 + 1$	1-1/4 1 1-1/2			3 3 3	2 1-1/2 2-1/4	9 9 9+1			
									First	Gr	oup To	tal	5	117 + 3			
Single Lap	[0] <sub>c</sub> [0/90] <sub>c</sub> [0/±45] <sub>c</sub>	3 	1/4  1/2	3 3	1/4 - 1/2	3 	1-1/4 - 1-1/2	3 = 3 + 1	1-1/4  1-1/2	3 - 3	2-1/4 - 2-1/2	3	2-1/4 	18 0 18 + 1			
Double Lap	[0] <sub>c</sub> [0/90] <sub>c</sub> [0/±45] <sub>c</sub>	3 3 3	1/4 1/4 1/4	3 3 3	1/4 1/4 1/4	3 3 3	1/2 3/4 1	3 3 3 + 1	1/2 3/4 I	3 3 3	1 1-1/2 1-3/4	3 3 3	1 1-1/2 1-3/4	18 18 18 + 1	MB-329(HS-LE) adhesive		
Step Lap	[0] <sub>c</sub> [0/90] <sub>c</sub> [0/±45] <sub>c</sub>	1	· 11	יי איז איז	1/2 1/4 1/4		-	3 3 3 + 1	1-1/2 3/4 1-1/4	1 1 1	-	3 3 3	2-1/4 i-1/2 2-1/2	9 9 9 + 1			
								S	econd	Gro	oup To	tals		117 + 3			
								C	ROUP	TC	TALS			234 + 6			

\* Adherend Material: B-B is ooron-to-boron; B-T is boron-to-titanium; N = Number of Specimens; O.L. = Overlap Length in inches.

(a) All simple specimens in this column will have clamp-on (2-in, gage length) extensioneter used to record detormation during loading.

(b) All "special" specimens in this column will be used to determine load deformation behavior of joint (e.g., strain gage instrumentation).

# SECTION VI

# LAMINATE PROCESSING

# VI.1. GENERAL

The purpose of this section is to describe the processing, fabrication and quality control activities in making composite laminate panels during this research program. These are presented in Sections VI.2, VI.3 and VI.4. Processing Facilities described in Section VI.2 give detailed information about the type of buildings and equipment used, while Section VI.3 on Process Development provides the reader with a brief summary of the developmental aspects of the materials processing. Adherend Panel Fabrication and Quality Control are presented in Section VI.4 and cover the processing and inspection details of panels used in the experimental program.

# VI.2. PROCESSING FACILITIES

Special facilities and equipment are required for composite fabrication and quality control. This section describes these areas.

The processing laboratory\* in which cleaning and layup of the boron/epoxy and fiber glass/epoxy laminates and the bonded joints was accomplished is  $19 \times 20$  feet with air conditioning supplied by the zone-controlled central building unit. During a normal week, the temperature in the laboratory varies between  $72^{\circ}$  and  $74^{\circ}$ F while relative humidity varies from 51 to 57 percent. Temperature and humidity in the laboratory were recorded continuously by a Honeywell two-pen recorder actuated by a mercury-filled temperature sensor and a hair humidity sensor. Extremes recorded during the period of this tesearch were  $05^{\circ}$  to  $75^{\circ}$ F in temperature and 40 to 05 percent relative humidity.

Equipment in the laboratory included a work bench, several work tables (two with Formica® tops for cleaning operations and layup), a Formica®-topped wash basin made of special chemical resistant molded epoxy reinforced plastic, storage cabinets, an air circulating Blue M oven capable of controlled temperatures up to  $500^{\circ}F \pm 2^{\circ}F$ , and a clean type deep freeze for storage of preliminging direct insterials at  $0^{\circ}F$ 

The 50-ton M and N press is located in an adjacent laboratory (same building) which is also air conditioned, but the temperature and humidity may vary more widely since it is a large open area with direct access to the outside. Figure 53 consists of photographs of the laboratory and associated equipment.

## VI.3. PROCESS DEVELOPMENT

The development of a standard process (see Appendix C) for making laminates and inspecting them was required to provide the necessary consistency and control for the adherend materials to be used in the experimental effort. While the processing and laminating variations were investigated, so was equipment functioning. Besides the hand lavup process, the two main areas of concern were the laminating press and the thru-scan ultrasonic inspection system.

Providing a laminating press which had closely controlled temperatures was the first order of business. After considerable overhaul and modification of the 50-ton M and N press with  $20 \times 24$ -inch electrically heated platens the following heat survey/adjustment procedure was begun.

Separate recorder-controllers were connected to the contactors of the top and bottom platens. Four 20  $\times$  24-inch plates were cut from 0.125-inch aluminum. A slot from the center to one side was cut in the back side of each plate and a 26-gage iron-constantin thermocouple cemented into the slot with the hot junction at the center of

\*Located in the Department of Structural Research.



D. 50-Ton M&N Press

C. Strain Gage Work Area

FIGURE 53. COMPOSITE LABORATORY AND EQUIPMENT

the plate. These thermocouples serve as the control input to the platen temperature recorder-controllers. The face of each tool plate was sanded to remove scratches and given two coats of a wax base mold release agent.

An eight-ply, 16 × 20-inch heat survey panel (made from 1581 glass/5505 epoxy) was laid up with eighteen thermocouples arranged in a triangular pattern (see Figure 54) embedded between the fourth and fifth plies. Layup and cure of this panel was accomplished in accordance with the SwRI 03-301 Process Standard for Boron/Resin Composite Laminate Fabrication\*. During the cure cycle, the temperature at these thermocouples was monitored at frequent intervals. During the 200°F portion of the cycle, there was not more than 6°F difference between the highest and lowest thermocouple readings. There was an initial overrun of temperature on heat-up to 214°F which dropped within 15 min to 210°F, and was under control at 200° to 206°F during most of the remainder of the 2 hours. No overrun occurred at 300°F and control was maintained between 296°F and 302°F with a maximum difference of 12°F between the highest and lowest temperature readings. Control was maintained at 344° to 348°F with a maximum temperature difference of 14°F during the final 2 hours of cure. Figure 55 is a back-lighted photograph of the cured heat survey panel. The dark patches across the thermocouple wires are Scotch® Brand glass cloth electrical tape, No. 27, which was required to hold the thermocouples in place during layup and cure. The wires are 20-gage iron-constantin with enamel and glass fiber wrap on each wire and glass braid over all. Quality of the panel was visually good with a generally uniform light yellow translucent appearance.

Fifteen panels were fabricated using the 1581 glass fabric/5505 epoxy material and eleven panels were fabricated from Narmco 5505 boron/epoxy Lot No. 297, Roll 13 (twisted fiber) and Lot No. 373, Roll 1, which was the first production lot of material received. These panels are listed in Table III.

Panel No. 6 has pieces of Teflon®-coated glass fabric and Scotch® Brand glass cloth electrical tape, No. 27, embedded between the fourth and fifth plies as shown in Figure 56. Figure 57 is a back-lighted photograph of this panel. This was used to develop the ultrasonic test technique for voids and inclusions.

Initial laminates using the boron prepreg had rather poor appearance. The top surface was resin-starved. Panels B-1 and B-2 were cured at a higher total pressure than had been used on an equivalent size of glass fabric reinforced panels, but the increased pressure was evidently not sufficient to accomplish the greater compression of the Coroprene boundary support required by the thinner boron layup. A 0.020-in, aluminum shim was placed under the layup for Panel B-3, and the press was adjusted to the load used previously for the glass fabric/epoxy panels. This also resulted in a resin-starved surface. An increased load was then used for panels B-4 and B-5. This improved the resin flow and, except for a few loose fibers in one location on the surface of B-4, the appearance was good. The later boron laminates all had good appearance, except Panel B-9 which had some loose fibers on the top surface.

Tensile strength specimens were prepared from Panel G-2 and were tested on the Instron machine at a constant deflection rate of 0.05 in./min. The average ultimate tensile strength of nine specimens was 56,700 psi, with a proportional limit of 30,030 psi. The average modulus was  $4.78 \times 10^6$  psi initially and  $3.17 \times 10^6$  psi above the proportional limit. Complete data on these tests are presented in Table IV. While these tests were performed for the IRAD Creep Program<sup>+</sup>, they provide an indication of the quality of the fabrication technique.

A through transmission ultrasonic inspection facility was completed and all glass/epoxy panels, except G-1, 2 and 5 and boron/epoxy panels through B-8, were subjected to ultrasonic inspection. Panel G-1 is the heat survey panel with thermocouples imbedded as shown in Figures 54 and 55. Panels G-2 and G-5 had already been cut for test specimens; however, the larger remaining pieces of these panels were inspected. The recording of the ultrasonic test of Panel G-6 is shown in Figure 58. Plastic tape was placed on all edges and the whole panels were sprayed with clear lacquer to prevent water absorption while immersed in the water bath. The edge tape shows as pips along each end of the panel and solid lines along one side. Each line on the chart represents a 1/8-inch interval on the panel.

# \*Appendix C, Page C-9.

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SwRI In-House Research and Development Program, Project 03-9036.



 $\oplus_{\operatorname{Survey Thermocouples}}$ 





FIGURE 55. HEAT SURVEY PANEL-BACK-LIGHTED PHOTOGRAPH

A. 158	1 Glass Fab	ric/N-5505	5 Epoxy Pane	ls	
Panet No.	Size, in.	No, of Plies	Thickness, in,	Fiber Orientation	Remarks
G-1	16 × 20	8		0°	Heat Survey Panel-18 Thermo- couples at Center
G-2*	16 × 20	8	0.072	0°	To be used for Tensile Test Specimens
G-3*	16 × 20	12	0.103	0°	To be used for Flexural Test Specimens
G-4*	6 × 10	8	0.067	0°	
G-5*	6 × 10	12		0°	
G-6*	6 × 10	8		0°	Void and Inclusion Test Panel
G-7*	16 × 20	10	0.090	0°	
G-8*	16 × 20	9		0° , 90° ,, 0°	
G-9*	6 × 10	11		±45°	
G-10*	6 × 10	19		0°, ±45°,, 0°	
G-11*	16 × 20	11	0.0975	0°	
G-12*	16 × 20	11	0.0960	90° to long axis of panel	
G-1 3*	16 × 20	11	0.0940	±45°	
G-14*	16 × 20	11	0.0930	0°, 90°,, 0°	
G-15*	16 × 20	13	0.115	$0^{\circ}, \pm 45^{\circ}, \ldots, 0^{\circ}$	
B. Boro	on Fiber/N-5	505 Epox	y Panels		
B-1*	6 X 9	9	0.0493	0°	Surface is resin starved
B-2*	6 X 9	8	0.0450	0°	Surface is resin starved
B-3*	6 x 10	8	0.0357	0°	Surface is resin starved
B-4*	6 X 9	9	0.0458	0°, 90°,, 0°	Loose fiber on one surface
B-5*	6 × 10	9	0.0465	±45°	Good appearance
B-6	6 × 9	9	0.0474	0°	Used new material from Lot 373
B-7	6 × 10	15	0.0791	0°	Used for acceptance tests of Narmco 5505 Lot 373, Roll No. 1
B-8	6 X 9	17	0.0784	0°, 90°,, 0°	From Lot 373, Roll No. 1
B-9*	6 × 10	8	0.0447	0°, ±45°,, 0°	Loose fibers on one surface
B-10*	6 × 9	15	0.0757	90" to long axis of panel	
B-1 i *	6 × 9	17	0.0803	0°, 90°,, 0°	
+1RAD	Panels for Ci	reen Progr	am.		

TABLE III



FIGURE 56. PANEL NO. G-6 WITH TEFLON-COATED GLASS FABRIC AND GLASS CLOTH ELECTRICAL TAPE INCLUSIONS FOR ULTRASONIC TEST CALIBRATION



FIGURE 57. PANEL NO. G-6-BACK-LIGHTED PHOTOGRAPH

# TABLE IV

Dumension of Property				Spec	imen Num	bers				Average
	-1-1	-1-2	-1-3	-2-1	-2-2	-2-3	-3-1	-3-2	-3-3	
Gage Dimensions: Length (in.) Width (in.) Thickness (in.)	2.00 0.501 0.070	2.00 0.501 0.069	2.00 0.501 0.068	2.00 0.502 0.069	2.00 0.502 0.069	2.00 0.500 0.069	2.00 0.501 0.069	2.00 0.500 0.069	2.00 0.497 0.070	
Proportional Limit Load (lb)	1,010	950	980	940	1,140	1,090	1,050	1,120	1,080	
Maximum Load (Failure) (lb)	2,050	2,025	1,950	1,860	1,775	2,000	2,115	1,940	1,950	
Ultimate Tensile Strength (psi)	58,300	58,600	57, <b>2</b> 00	53,800	51,200	57,900	61,200	56,200	56,100	56,720
P. L. Strength (psi)	28,750	27,400	28,700	27,100	32,900	31,600	30,300	32,500	31,000	30,030
Initial Modulus (× 10 <sup>-6</sup> psi)	4.08	4.74	4.88	4.92	4.48	4.78	4.55	5.81	4.78	4.78
Final Modulus (X 10 <sup>-6</sup> psi)	3.12	3.14	3.44	3.68	3.04	2.82	2.88	2.75	3.65	3.17
*Performed for IRAD Creep Program	by SwRI									

# **TENSILE STRENGTH TESTS\* OF PANEL G-2**

The tape inclusions in the panel are shown distinctly but are not very well defined by size or shape. Ultrasonic tests of Panels B-2 and B-8 show extensive areas of reduced ultrasonic transmission. Panel B-8 is a  $0/90^{\circ}$  layup, and the tape pattern is apparent in the chart (Fig. 59). Panel B-2 is a  $0^{\circ}$  layup, and no particular pattern is presented on the chart (Fig. 60). These two results indicate that the recording system's sensitivity was too high during this test. Panel B-7 was an acceptance test panel and is shown in Figure 61 at a lower sensitivity.

Areas of reduced transmission also appear in one corner of Panels G-7, G-11, G-12, G-13, G-14 and G-15. It was possible to determine the significance of these areas by cutting flexure specimens which included these areas of reduced ultrasonic transmission. A more detailed study of Panel No. G-11 will reveal this.

Typical and reduced performance of such flexure specimens is illustrated by the results of tests on glass fabric/ epoxy composite (Panel No, G-11). A data summary package is included here as Table V and Figures 62, 63, 64 and 65. Table V gives general information on the prepreg material and the cured laminated panel. The fiber orientation indicates the warp direction of the 1581 style woven glass fabric. The 2387 epoxy resin system is the same that is used in Narmeo's Rigidite 5505 boron/epoxy materials which were used as the primary adherend materials in this program. The material was 2 months beyond the warranty expiration date when cured, but it does not appear to have deteriorated to any significant extent.

Figures 62 and 63 represent the average results of flexure tests on three specimens each from adjacent areas of the panel in which the ultrasonic inspection indicated no flaws and extensive flaws, respectively. The flawed area flexure strength and modulus were slightly lower than in the area with no flaws. Figure 64 is the cutting pattern for Panel G-11 in approximately true proportion. Figure 65 is the ultrasonic inspection record for Panel G-11. This is not in true proportion to the panel. The long dimension of the chart represents the 16-in, width of the panel. Each line across the chart represents 1/4 in, in 18-1/2 in, of the 20-in, length of the panel (approximately 1-1/2 in, at the end of the panel, area 11-3 in Fig. 64, was not inspected). The portions of the panel used in these flexure tests are outlined in Figure 65. The other specimens cut were for the IRAD\* program only.



#### TABLE V

# PANEL DATA SUMMARY PACKAGE PANEL NO. G-11

MATCHIAL INCOMATION

١.	MATERIAL INFORMATION	evide
А	Material Type: Narmco 2387-1581-38	3-in. evide
В.	Date of Manufacture: 6-10-69	layu foun
C.	Material Confirms to Specification:	areas the p
D.	Prepreg Resin Content (Volume %): 34.0	fiber remo
E.	Batch No.: 11	acce
F.	Roll No.: 1	epox
G.	Warranty Expiration: 9-11-69	Pane 1969
LAN	MINATE INFORMATION:	to ul 68).
А.	Orientation: [0] 11T	tudir tal sł
В.	Process Record No.: G-11	given The s
C.	Process Conforms to Specification:	test r tion
Đ.	Cure Date: 11-14-69	slight
E.	Number of Plies: 11	VI.4.
F.	Average Panel Thickness (in.): 0.0975	
G.	Average Ply Thickness (in.): 0.00886	ricate one a
Н.	Fiber Content (wt %): 70.810	with thick
1.	Resin Content (wt %): 29.190	meas perin
J.	Void Volume (Volume %): 3.87	meas 0.000
		Panel

- K. Panel Density (lb/in.<sup>3</sup>): 0.0671
- L. Panel Size:  $16'' \times 20''$

П.

The boron panels were also X-rayed. A positive print of Panel B-4 is shown in Figure 66 and Panel B-8 in Figure 67. These are both  $0^{\circ}/90^{\circ}$  layups, but B-4 was prepared with material from Lot 297, Roll 13 and is nine plies thick, while B-8 was made from Lot 373. Roll 1 and is seventeen plies thick. Spaces are quite ent between the 1/8-in. tapes used in preparing the wide tapes in each of these panels. Panel B-8 shows ence of poor spacing between the 3-in, tapes during p of the panel which also corresponds to the pattern d in the ultrasonic inspection. The fiber-poor (light) on Panel B-4 running parallel to the long axis of banel (along the edges) are the areas where surface s were loose and peeled off when the panel was oved from the tool plate. Panel B-7, the material ptance test panel, is shown in Figure 68.

Rolls 1, 2 and 3 of Lot 373, Narmeo 5505 boron/ epoxy prepreg were received on December 16, 1969. Panel B-7 was prepared from Roll 1 on December 23, 1969, for acceptance testing. This was first subjected to ultrasonic and X-ray examination (see Figs. 61 and 68). The panel was then cut into specimens for longitudinal and transverse flexure strength tests and horizontal shear strength test. The results of these tests are given in Table VI along with the results from Narmeo. The specimens after test are shown in Figure 69. The test results all satisfy the General Dynamics specification FMS-2001 except the flexural modulus which is slightly (5%) low.

# VI.4. ADHEREND PANEL FABRICATION AND QUALITY CONTROL

All composite adherend material panels were fabricated in accordance with Figure 70. These panels and one acceptance test panel (B-20) are listed in Table VII with the ply thickness and fiber orientation. The panel thicknesses shown represent the average of a number of measurements taken 1 in. from the edge around the perimeter of the panel. Except for Panel B-20, these measurements indicate a ply thickness value of 0.0053  $\pm$ 0.0004 in. average for the boron/epoxy composites. Panel B-20 is indicated to have a ply thickness of 0.00486 in.; however, measurement of the ply thickness in a microphotograph gave a value of 0.00518 inch.

Panel B-20 was a  $6 \times 10$ -in, panel which was prepared for acceptance testing of the Narmeo 5505 mate-

rial from Batch No. 381. This shipment was comprised of Roll Nos. 30, 31, 32, 33 and 73, which were received on March 5, 1970. The ultrasonic test chart of this panel is shown in Figure 71, and a positive print of the X-ray is shown in Figure 72. The results of the longitudinal and transverse flexure strength and modulus and horizontal shear strength tests are given in Table VIII along with the qualification test results from Narmoo and the General Dynamics 4 MS-2001 specification requirements. All acceptance test results were substantially in excess of this specification's requirements and also exceeded the Narmoo qualification flexure values.



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FIGURE 65. ULTRASONIC THRU-SCAN RECORD-PANEL G-11





FIGURE 66. X-RAY OF PANEL B4



FIGURE 67. X-RAY OF PANEL B-8



FIGURE 68. X-RAY OF PANEL B-7

# TABLE VI

# MATERIAL ACCEPTANCE TESTS

Na	rmeo 5505 B	oron/Epoxy I	Prepreg Lot 373	, Roll 1							
	Lamii	nate Fiber Or	ientation: 0°								
	Load Orientation: $0^{\circ}$ and $90^{\circ}$										
Laminate Thickness: 15 plies - 0.080 inch											
Discoi est. Deservatur	Test	SwRI Acco	eptance Tests	Narmeo	G.D.'s Spec.						
Physical Property	Temp, ° F	Spec. No.	Results	QC Report	FMS-2001						
Flexural Strength-	RT	B-7-1	233,260								
Longitudinal (psi)	RT	B-7-2	268,130								
	RT	B-7-3	234,490								
		Average	245,290	225,900	225,000						
Flexural Modulus-	RT	B-7-1	$29.85 \times 10^{6}$								
Longitudinal (psi)	RT	B-7-2	27.65 × 10 <sup>6</sup>								
	RT	B-7-3	$27.85 \times 10^{6}$								
		Average	$28.45 \times 10^{6}$		$30 \times 10^6$						
Flexural Strength-	RT	B-7-4	14,770								
Transverse (psi)	RT	B-7-5	9,375								
•	RT	B-7-6	12,300								
		Average	12,150	14,200	10,000						
Horizontal Shear	RT	B-7-7	15,390								
Strength (psi)	RT	B-7-8	15,510	-							
B (L)	RT	B-7-9	15,330								
	RT	B-7-10	15,340								
		Average	15,390	13,400	13,000						



FIGURE 69. MATERIAL ACCEPTANCE TEST SPECIMENS



# FIGURE 70. PANEL DRAWING BONDED JOINT STUDY

# TABLE VII

# PANEL FABRICATION FOR SIMPLE BONDED JOINT EVALUATION

Panel	Nominal	No of	Measured Average Panel	ed anel Filter 55. Orientation*	Mate	rul ssos	Remarks	
No	in	Plies	Thickness, in	Orientation*	Batch No	Roll No	Remarks	
B-12	16 × 20	4	0.01655	[0] 17	373	1		
B-13	16 × 20	6	J 03325	101	173	Т		
B-14	16 × 20	4	0.04918	{{0}/90}₄/0] <i>T</i>	17 1	1		
B-15	16 # 20	17	0.09098	[(0/901 <sub>87</sub> 0] 7	173	1,2		
B-16	16 × 20	9	0.04925	[0/+45/0/=45/0] <sub>3</sub>	373	2		
B-17	16 × 20	17	0.09150	[(0/+45/0/=45)g/0]3	173	4		
8-1×	16 × 20	6	0.03167	101	173	3		
B 19	16 × 20	17	0 0°KU 77	[(0/90) <sub>6</sub> /0] <sub>1</sub>	17.3	1		
B-20	6 ₹ 10	15	0.07290	[0]	381	10	Acceptance Test Panel	
B-21	16 + 20	9	0.04760	10] 97	5 <b>H</b> I	ю		
B 22	16×20	17	0.09117	{(0+15/0-45) <sub>Q</sub> /0] <sub>A</sub>	38.1	30, 31		
B 23	16 + 20	16	U Oh Hibh	[0.90, 0] + 1	181	30, 31		
B 24	16 × 20	9	0.04835	[0/+45.0/=45/0] <u>s</u>	₹ні	10 H		
B-25	16> 20	×	0.04126	[0] x z	181	н		
B-26	16 + 20	16	0.08485	[(0+1540g] v	181	4.2		
B 27	16 × 20	16	0.08460	1 10 145 0ig [s	381	32, 33, 73		
8.28	16 + 26	9	0.04494	[(0+45.0)]45.0]3	าหา	74		
B 29	6+ 9	h	014610	0/907-0127	UK I	23 	Void Standard Punel	
B 40	6+9	Fe	0.07*10	j0 %0 <sub>2</sub> 0] 42	38	1	V od Standard Panel	



FIGURE 71. ULTRASONIC INSPECTION RECORD FOR PANEL B-20

# TABLE VIII

# MATERIAL ACCEPTANCE TEST

#### March 20, 1970 oy R. E. Tuck

Natineo 5508 Boron Epoxy Prepreg Lot 381, Roll 30 Laminate Eiber Orientation / 0 Laminate Elinekness / 15 ph/s. / 0.073 m

Physical Property	Lest Temp, F	SwRI Spec No	Results	Natineo QC Report	G D % Spec 1 MS 2001
Hexural Strength	RI	B.20-4	258 772		
Longitudinal	RT	8.20.5	280,430		
(psi)	1.1	B 20.6	277.181		1
		Average	272,128	245,100	225,000
Hexural Modulus	RT	8.20.4	30 '67 10'		
1 ongitudusal	K I	B 20 1	] 31 49 × 10 <sup>0</sup>		
(1951)	RT	B 20 6	32.51 < 101		1
	1	Average	31.60 - 101		30 < 10*
Eleviral Strength	R	B 10-1	13.145		
Transverse (psi)	RI	B.20.2	15 15		
	RT	B.203	11604	ļ	
		Average	1.4 SD3	14100	10 000
Eleveral Modulus	RI	B-20-1	196 + 10'		
Lanverse (par	K I	в.0.2	$1.261 \times 10^{6}$		ļ
	RT	B 20 3	≥ 64 × 10 <sup>4</sup>		
	Ì	Average	2 2 4 10		
Honzental Shca	RI	B.20	14.225		
Strength (psu	RI	B 20 %	14-62		1
	RI	B 10.9	119.5		
	RT	B_0.0	H D		
		Asies	1.1 -1.	1 5 GUD	1 I I I I I I I I I I I I I I I I I I I



FIGURE 72. POSITIVE PRINT OF X-RAY OF PANEL B 20 USED FOR ACCEPTANCE TESTING

Panels B-29 and B-30 were manufactured with known void inclusions for use as void standards in comparisons with the actual panels. A scale drawing of the void inclusion panels is shown in Figure 73, locating and describing



# FIGURE 73. BORON REINFORCED ULTRASONIC TEST PANELS

them precisely. The ultrasonic record of void panel B-29 is shown in Figure 74, and that of void panel B-30, in Figure 75. Two void panels, one 8-ply and one 16-ply, were necessary to adjust the ultrasonic inspection system. Most of the panels used in the program are close to these numbers of plies.

The ultrasonic inspection recorder charts (see Figs. 74 and 75) for the  $6 \times 9$ -in. panels, B-29 (8 plies) and B-30 (16 plies), contain the following built-in voids. Pieces of TX-1040 Teflon®\*-treated glass fabric, 0.001-in. thick, were placed between the center plies. The top row is composed of square shapes ranging from 1/8 to 1 in, in size. The second row is composed of circles of the same diameters. The third row, from left to right, contains  $1/8 \times 1$ -in. long strips spaced 1/8, 1/4 and 3/8 in. apart, oriented at 0° to the direction of the ultrasonic scan. Next are strips oriented at 90° to the scan direction. The first strip is 1/4 in. wide and the others are 1/8 in. wide and again spaced at intervals of 3/8, 1/4 and 1/8 inch. These are followed by a 1/8- and a 1/4-in. wide strip at a 45° angle. Below the 1/4-in, wide, 45° strip, a 1/8-in, wide  $\times$  i-in, long strip of boron/epoxy prepreg was placed between the center plies. Below the 1/8-in., 45° strip, three strips of boron/epoxy were placed between plies 2 and 3, 4 and 5, and 6 and 7. The boron/epoxy strips were also oriented at 45° to the scan direction. A 0/90 cross-ply fiber orientation was used for both panels.

The 1/8-in. diameter circle was lost at the 6-dB sensitivity level required to minimize extraneous signals in the 16-ply panel (B-30). Some other areas of apparent thickness discontinuity are present in addition to the TX-1040 fabric. The extra layer of boron/epoxy also is detected by the ultrasonic test. In the 8-ply panel (B-29) the 1/8-in. square and circle are barely detectable at a 4-dB sensitivity. The single extra ply of boron/epoxy shows up more

\*Registered trademark, E. I. DuPont de Nemours.



FIGURE 75. ULTRASONIC INSPECTION OF VOID PANEL B-30

readily than in the 16-ply panel. Some unplanned apparent thickness discontinuity areas are also apparent in this panel. The ultrasonic thru-scan and radiograph inspection records for adherend panels B-12 through B-28 are shown in Appendix D.

Marking and machining of the  $16 \times 20$ -in, boron composite adherend panels into lap shear assembly details is shown in Figure 76. A 15/16-in, wide strip was cut from the long edges of each panel. Two 9-in, long tensile test specimens were cut from each strip (four per panel). These had glass-fabric/epoxy (1581/5505) load pads bonded (with AF 126-2) to the ends for the monotonically loaded tension test which provided the complete stress-strain curve for strength and modulus determination of each panel\*. Standard constituent properties of boron/epoxy laminates are presented in Appendix B.

Fiber content of each panel was determined by the fiber count method using from two to six specimens cut at each end, and the center of each of the strips to be used for tensile test specimens. Figure 76 shows the location of the strips and individual specimens relative to the panel and adherend pieces







\*See Appendix C for Test Method, Appendix E for typical data.

# SECTION VII

# LAMINATE AND TITANIUM ADHEREND TEST RESULTS

# VII.1. GENERAL

Section VII is devoted to summarizing and discussing the results obtained from testing composite and titanium adherend materials. Section VII.2 analyzes the Laminate Adherend Experimental Results whereas Section VII.3 summarizes the Titanium Adherend Experimental Results.

# VII.2. LAMINATE ADHEREND EXPERIMENTAL RESULTS

Selected typical tensile test data on the N-5505 boron/epoxy laminates are given in Appendix E. There are typical stress-strain curves and cross-section photomicrograp is presented for each different material batch/orientation group of four tensile specimens. A summary of the key properties has been made from the detailed data and is shown in Table 1X. Typical, representative failed specimens are covered in Figures 77 and 78.

# VII.2.a. Laminate Performance

It becomes obvious after study of this data that laminate tensile specimen performance was not up to par with 2nd Edition Design Guide data.\* In an attempt to establish the magnitude of the discrepancy from the normal or expected values the unidirectional data were analyzed. Panels B-12, 13 and 18 from material Batch No. 373 and Panels B-21 and 25 from material Batch No. 381 were  $[0]_{nT}$  laminates from which tensile specimens and adherend materials were cut with their longitudinal axis parallel to the fibers. The method used for properties prediction was originally proposed by Tsai<sup>(17)</sup> based on the "rule of mixtures" technique. The "k" (and k') factor used by Tsai was called a fiber misalignment factor. In the analysis here it shall be called the "void factor" and based on an empirically developed mathematical form of the laminate decimal void-volume (V.V.). The results of this study along with the formulas are presented in Table X. These formulas give results which are strongly dependent on the fiber volume (F.V.). In addition, the factor  $K_0$  (and  $K'_0$ ) is a function of the matrix/fiber modulus or strength ratio. Here another deviation was used. Both the matrix modulus and strength were utilized as being the average of the matrix material and a 104 glass scrim laminate.<sup>+</sup> Since the tensile stress-strain curve of this boron/epoxy material is linearly elastic<sup>+</sup> almost to failure it was felt satisfactory to use the rule of mixtures technique to predict ultimate strength as well as modulus.

A study of the photomicrographs of Appendix E will reveal that all the reinforcing fibers in the laminates were badly cracked but fully encased in resin throughout the crack spaces. Therefore, it must be assumed that the fibers were cracked prior to laminating the prepreg material in the hot platen press. It also is assumed that the extensive amount of cracked and broken fibers visible in the photomicrographs had not occurred at the time of impregnating them with resin. This assumption is based on the judgment that (micrographs cracked and broken fibers would have been impossible by present methods. There is the possibility of course that the breakage could have occurred immediately after impregnating due to handling of the prepreg manufacturer or due to subsequent handling by the laminator.\*\* In the author's judgment the latter two possibilities are remote

<sup>†</sup>See Appendix B.

‡Sometimes in two stages.

\*1SwRL

 $<sup>*</sup>F_L^{\eta i} = 188 \text{ ksi}, E_L^i = 30 \times 10^6 \text{ psi}.$ 

TABLE IX

and the second

IL STREET

LAMINATE MECHANICAL/PHYSICAL PROPERTIES DATA SUMMARY

							2.7	SOS Horne / Par	Daniel						
Panel Number	B-12	B-13	B-14	B-15	B-16	8-17	B-18	B-19	B.71	с. н. 17			2,4		
Parameters	[0]3T	[0] 6T	[(0/30)4/0]	T [0 \$(06/0)]	[0 +45'0' 45 G]S	101 +45 0 4510 01S	[0] hT	$T[0]_8(00,0)]$	[4] v]	1 5 0 0 15 0 5t+101	0 402/0] 4T	2[0.57 0.57+.0]	18 [0]	8-20 [10/:45 10/0]S	9 -45 0/ 45/0]S
No. of Plies	3	\$	•	1	6	-	¢	<u>15</u>	σ	17	16	•	~	16	σ
au. ksi	122.539	146 738	59 972	>2876	9t U 36	640 88	142 726	-\$ 110	- 82 1	109 605	500 05	062 66	177 850	93.524	93.989
นเ/นาท เ⊓∍	4,727	5,382	3,658	3.577	5,110	5,558	015.2	3,622	6.068	9.246	3.140	6 182	6.255	5,614	5,342
u1, u1, vu	964	1,145	129	104	3,435	3 665	65U'1	10	1.208	4.538	86	4.129	1.391	4.020	3,548
opt. ks:	97 753	127.960	34 861	27.4-6	61 824	1.5 +9	120.013	33.012	174 544	fac 5ac	217 PC	73 155	166 735	u61'12	77.930
u//um/13d2	3, 556	4,638	1,902	269'1	3,355	+ ¥.	4 282	076.1	5.944	3, 2, 15	1.626	4.333	5.808	4.186	FÜF'F
u1/-и1т - 2 3d э	744	975	85	65	2,205	2.5 0	124	r c	1.190	2.512	60	2.929	1.739	2.975	2,952
Ep. kksi	26 767	27 682	18 095	15 967	18 435	16 540	28 312	17 (41	24 535	4,781	15 356	16.873	28 701	17.027	17 705
Es, kksi	22 062	.5 314	14 390	13.113	16 44()	14.548		13 365		16.832		15 654		15 654	
4	0.2134	0 2087	0.0427	0 0395	0.6578	8234 0	5102.0	0.0342	9561.0	0 7212	01200	0 6025	2112.0	5612.0	0.6790
5	0 2165	0 2065	0.0334	0 330	0 6466	0.6503		0180.0		11110		0 6648		0.7160	I
F.V. Fraction	0.536	0 498	0.476	9.508	() 468	0.520	105 ,	0.512	0.520	96F U	\$05 O	0.506	0.505	0 484	0.495
V.V Fraction	0 160	0.014	0 004	c	¢	0	070	c	0.018	¢	c	¢	¢	•	0
Density, Ib/m. <sup>3</sup>	0.0643	0.0688	9.0681	0.0710	0.0583	dt-20.0	0.0678	2120.0	- 690 0	11200	0.0712	+0_0 Q	0.0698	0.0713	0.0721
% Cracked Filaments	94.5	96.0	898	94 ()	93.0	v *	80.0	\$ 0.4	5 5	e S	017	5			0.60
Thickness per ply.in	0.00551	0.00554	0.00546	0.00535	0 00547	2 5 2 (0) ()	0.00528	<b>1</b> 1 SIND II	80,000	4 605 36	61500 v	- E SIQ ()	0.01516	115400	
Total Thickness, in.	0.01655	0 03325	0.04918	86060.0	0.04925	05160.0	0.03167	2,000-0	0.047.60.03	- 1160 ti	0 08 308	25.840.0	<b>1</b>	00	
Material Batch/ Lot	373/1	373/1	373/1	373/1 & 2	373/2	3/3 3	373.3	E 64E	381/30	381/30 & 31	18 30 & 31	181 3.0 2.181	15/185		



# TABLE X

Batch	Panel No.	Panel F.V.	Panel V.V.	Panel Density, 15/in. <sup>3</sup>	Exper. <i>E<sub>p</sub></i> , 10 <sup>6</sup> psi	Calc. $E_{\varrho}$ , $10^6$ psi	$\frac{\frac{E_p}{E_q} \times 100}{\%}$	Exper. <i>o<sub>u</sub></i> , ksi	Cale. F <sub>tu</sub> , ksi	$\frac{\sigma_u}{F_{u}} \times 100,$		
	B-12	0.536	0.160	0.0643	26.767	31.140	85.96	122,539	168.739	72,62		
373	B-13	0.498	0 014	0.0688	27.682	29,752	93.04	146,738	184.934	79.34		
	B-18	0.501	0.040	0.0678	28.312	29,938	94.57	147,726	181.090	81.58		
381	B-21	0.520	0.018	0.0697	29,535	31.021	95.21	177.387	191.608	92.58		
	B-25	0.505	0	0.0698	28.701	30.218	94,98	177,856	190.080	93,57		
		and hereits a success of disease a star dama			Form	ulas						
	Modulus Strength											
-	Ŀ	$   = kE_f $	[] K <sub>0</sub> (	i F⊻.)]	ŀ	$T_{tu} = k' F_f [1]$	$K'_0(1$	Ē.V.)]				
	k	= ] (\	$(N_{*})^{2}$		K	k' = 0.80(1 - V.V.)						
	$K_0 = 1 - \frac{E_m}{E_f} = 0.9682$ $K'_0 = 1 - \frac{E_m}{E_f} = 0.9534$											
	Properties											
	Ŀ	$m = \frac{(3.2)}{3}$	+ 0,487	) × 10 <sup>6</sup> = 1	1.844 × 10	)" ря	$E_f =$	$58.0 \times 10^{6}$	ры			
	ŀ	$\frac{37.8}{m} = \frac{37.8}{37.8}$	3 + 4.184 2	= 20,992	ksi		$F_f = $	450,0 ksī				

# UNIDIRECTIONAL LAMINATE PERFORMANCE

because (1) handling the material after impregnating does not usually involve severe mechanical impingements and (2) making flat laminates does not normally induce such damage. Since this fiber was one of the first batches delivered after the twisted fiber episode with the fiber makers was solved, the tiber manufacture appears to be at fault. It is the authors' judgment that built-in residual stresses in the boron fibers caused cracking and breaking at some time after impregnation but before curing.

What is amazing about this problem is that only the tensile tests and photomicrographs uncovered the phenomena and the flexure tests and ultrasonic and radiograph inspections on the cured panels did not reveal the problem.

For Material Batch 373 in Table X it can be seen that Panel B-12 had a very high void volume (16%) with a resultant substantial reduction in the tensile properties from that of the low void volume laminate of Panels B-13 and B-18. Panel B-12 exhibited properties realization percentages of 85.96% for modulus of elasticity and 72.62% for strength whereas the average of Panels B-13 and B-18 gave 93.80% for modulus and 80.46% for strength. This illustrates the reduction caused by a high void content. Longitudinal flexure acceptance tests showed this material passed the minimum strength required (225 ksi) by 9% while the modulus failed by 5%. Transverse flexu, al strength was 42% above requirements (10 ksi) whereas interlaminar shear strength exceeded minimum requirements (13 ksi) by
only 3%. While the flexure acceptance test predicted closely what would happen to the tensile modulus the flexural strength was not even close as a predictor of tensile strength.

Material Batch 381, received later, was substantially better than 373 as indicated by both the acceptance tests and the tensile tests. However, the modulus and strength realization percentages\* are still only 95.10% and 93.08%, respectively, as shown by the average of Panels B-21 and B-25. This compares with longitudinal flexural strength and modulus values which exceeded the requirements by 21% and 5%, respectively. Transverse flexural strength exceeded that required by 41% whereas horizontal shear strength exceeded that required by 20%.

All this indicates that the flexure and interlaminar shear tests for acceptance are not good measures of cracked or broken fibers even though they may be good checks on impregnation and lamination processing variables.

Because of the fiber breakage problem, which was detected from photomicrographs on all panels regardless of orientation, all of the longitudinal tensile properties are somewhat low. However, it is doubtful that the transverse tension and shear tests of  $\{0\}_{nT}$  laminates would show any reduction, although transverse tests on the angleply ones would probably exhibit some degradation.

### VII.2.b. Laminate Orientation Sequence vs Performance

An interesting phenomena is the variation of properties with laminate orientation sequence. For the general crossply (0/90) orientation type the following comparison illustrates this point.

ltem	Panel Nos.	F.V.	Orientation	V.V.	<i>o</i> U, ksi	$E_P$ , $10^6$ psi	Batch
1	B-15/B-19	0.510	$[(0/90)_{8}/0]_{T}$	0	53,993	16.504	373
2	B-23	0.505	$[0/90_2/0]_{4T}$	0	40.005 13.988 Diff.	15.356 1.144 Diff.	381

The  $[0/90_2/0]_{4T}$  orientation exhibits a 26% reduction in strength and a 7% reduction in modulus compared to the  $[(0/90)_8/0]_T$  one. This occurred in spite of the fact that the  $[0/90_2/0]_{4T}$  orientation laminate was made from a substantially superior batch (381) of material. While the 12-1/2% more 0° plies in the first one over the second one may account for the modulus change it does not account for all of the strength change.

Another comparison can be made with the general  $0/\pm 45^\circ$  orientation as shown in the following table.

Item	Panel Nos.	<u>F.V.</u>	Orientation	<u>V.V</u> .	σ <sub>U</sub> , ksi	<i>Ep.</i> 10 <sup>6</sup> psi	Batch
1	B-16	0.468	$[0/+45/0/-45/0]_S$	0	90.036	18,435	373
2	B-24/B-28	0.500	[0/+45/0/-45/0]s	0	96.390 6.354 Diff.	17.289 1.146 Diff.	381
3	B-17	0.520	$[(0/+45/0/-45)_Q/\bar{0}]_S$	0	88.009	16.880	373
4	B-22	0.496	$[(0/+45/0/-45)_Q/\bar{0}]_S$	0	109.669 21.660 Diff.	18.476 1.596 Diff.	381
5	B-22	0.496	$[(0/+45/0/-45)_Q/\overline{0}]_S$	0	109.569	18.476	381
6	B-26	0.484	$[(0/\pm 45/0)_Q]_S$	0	93.524 16.145 Diff.	17.027 1.449 Diff.	381

\*As measured in tension.

In Items 1 and 2, identical nine-ply orientations are compared for the two different material batches. Items 3 and 4 compare identical seventeen ply orientations of the two different material batches. The strength values of Material Batch 381 show 7% and 25% improvement, respectively, over those of Batch 373 while the modulus values are about the same for the two pair of items. Comparison of the  $[(0/+45/0/-45)Q/0]_S$  orientation with the  $[(0/-45/0Q)]_S$  one is shown in Items 5 and 6. The first orientation shows an improvement of 17% in strength and 9% in modulus over the second one with both laminates being of the same batch of material. Again the strength increase is greater than would be indicated by the 12-1/2% increase in 0° plies whereas the modulus increase could be accounted for by this difference.

A direct comparison of Panels B-23, D-15/B-19, B-26, and B-22 is made below showing the trend of improvement with change of basic orientation and sequence.

Item	Panel Nos.	F.V.	Orientation	V.V.	σ <sub>U</sub> , ksi	<i>Ep</i> , 10 <sup>6</sup> psi	Batch
1	B-23	0.505	[0/90 <sub>2</sub> /0] <sub>4</sub> T	0	40.005	15.356	381
2	B-15/B-19	0.510	[(0/90) <sub>8</sub> /0] <sub>T</sub>	0	53.993	16.504	373
3	B-26	0.484	$[(0/\pm 45/0)_Q]_S$	0	93.524	17.027	381
4	B-22	0.496	$[(0/+45/0/-45)_Q/\bar{0}]_S$	0	109.669	18.476	381

From these comparisons it appears that the general  $0/\pm 45^{\circ}$  orientation is stronger and stiffer than the general  $0/90^{\circ}$  orientation and that further improvement can be made by separating either the plus and minus  $45^{\circ}$  plies or the  $90^{\circ}$  plies by  $0^{\circ}$  plies.

The orientations and thicknesses selected for adherend materials are behaved to be representative of those used in the acrospace indu. try. They cover ten orientation/thickness combinations in the fifteen  $16 \times 20$ -in flat panels made, from which adherends were cut for the bonded joint program.

### VII.3. TITANIUM ADHEREND EXPERIMENTAL RESULTS

The other adherend material was 6A!-4V annealed titanium sheet purchased from Titanium Metals Corp. (TMC). Four nominal thicknesses were used. These were 0.016, 0.032, 0.045 and 0.090 in. Two straight ided tensile specimens were tested from each thickness with the same equipment and instrumentation used to test the composite tensile specimens and the bonded joints. Average stress-strain curves for each thickness are contained in Appendix E. A summary of these properties is presented in Table XI whereas Figure 79 presents typical, representative tensile specimen failures. These average curves of Appendix E were used to obtain the Ramberg-Osgood parameters for use in the nonlinear analysis of bonded joints with at least one of the adherends made of titanium. Properties measured compared reasonably well with TMC and handbook typical properties. It can be observed from these data that the titanium becomes for more nonlinear than the composite materials studied in this bonded joint investigation.

### TABLE XI

### 6AL4V TITANIUM SHEET PROPERTIES SUMMARY

D		Nom.	Sheet	
Parameter	0.016	0.032	0.045	0.090
I <sub>ACT</sub> , in.	0.0165	0.0315	0.0455	0.0925
σ <sub>u</sub> , ksi	146.026	129.607	137.523	134.860
$\sigma_y$ , ksi	136.500	128.850	135.200	131.400
σ <sub>pk</sub> , ksi	116.834	125.356	123.584	112.700
$\nu_p$	0.3074	0.3086	0.3184	0.2902
$E_p \times 10^{-6}$	17.756	15.514	17.251	16.096
$\epsilon_{1y} \times 10^{-6}$ in./in.	9,575	10,225	9,750	9,985
$\varepsilon_{2y} \times 10^{-10}$ in./in.	2,700	=5,750	2,800	2,907
$\epsilon_{1p\ell} \times 10^{\ell}$ in./in.	6,590	8,074	7,171	7,004
$\epsilon_{2p\varrho} \times 10^6$ in./in.	2,012	-2,575	2,312	2,038
$\epsilon_{1u} \times 10^6$ in./in.	46,636	20,934	38,260	21,300
$\epsilon_{2u} \times 10^6$ in./m.	9,120	-7,885	14,362	:=3,735
TMC Typ. $F_{tu}$ , ksi	146.200	138.000	137.500	138.400
TMC-Heat No.	G-9075	G-9072	G-9520	K-3793
*Properties were ob configuration" tensi specimens was by dir	tained by u le specimen ect contact	use of stra one inch	ight sided " wide: gripp	composite ing tensile



FIGURE 79. TYPICAL FAIL: D Ti 6AI-4V ANNEALED TENSILE SPECIMENS

### SECTION VIII

### BONDED JOINT PROCESSING

### VIII.1. GENERAL

This section covers the processing and fabrication of the simple specimen bonded joints. Section VIII.2 presents the Adhesive Acceptance Test Results whereas Section VIII.3 covers the Simple Specimen Joint Fabrication.

### VIII.2. ADHESIVE ACCEPTANCE TEST RESULTS

A summary of the last acceptance test results for both the old and new batches of the two adhesive systems (AF-126 and MB-329) used in this program is presented in Tables XII and XIII. These tests were run on 29 October 1970. Earlier acceptance tests were run on the first Batch (724) of Scotchweld®\* AF-126-2 adhesive on 15 March 1970. The later Batch (739) of AF-126 adhesive was first tested on 24 July 1970. Initial acceptance tests on the Metlbond®† 329 adhesive batches were performed on (1) the first Batch 345/47 on 15 March 1970 and (2) the later Batch 360/40 on 24 July 1970. Batch 739 of AF-126 and Batch 360/40 of Metlbond 329 were used in fabricating the lap shear assemblies for this program. The data from these tables (XII and XIII) indicate that some degradation occurred with aging, however, all tests passed the MMM-A-132 specification requirements.

### VIII.3. SIMPLE SPECIMEN JOINT FABRICATION

Single and double lap shear assemblies were fabricated in accordance with Figures 80 and 81 covering both boron/epoxy to boron/epoxy and boron/epoxy to titanium joints with each adhesive system. A detailed listing of the single and double lap shear assemblies for single and double lap joints with boron/epoxy to boron/epoxy adherends utilizing both adhesive systems is shown in Table XIV. The boron/epoxy to titanium single and double lap shear assemblies with both adhesive systems are shown in Table XV. Fiberglass tabs were bonded on the boron/epoxy adherends with Eastman 910. Tabs were not used on the titanium adherends.

It was originally intended that three-step lap joints would be made by machining steps into the boron/epoxy laminates and titanium with subsequent matching and bonding as shown in Figure 82. Machining such steps in boron/epoxy proved nearly impossible with state-of-the-art diamond tools and cutting equipment. In consulting the Design Guide and several recent research investigations on machining boron/epoxy composites, nothing was found to guide our efforts on machining steps. After contacting the manufacturing experts of several aerospace companies it was found that most organizations lay-up and mold in the steps, usually in combination with bonding. However, several ideas were obtained on how step machining in boron/epoxy materials might be done. One idea was tried. Several diamond cup cutting wheels were tried with little success. After cutting the three steps in two different boron/epoxy laminate adherend materials and starting on a third, the tools were worn out. The manhour expenditure and the cutting tool wear rate on fabricating the steps that were made were orohibitive.

One machined step lap joint was made as shown in Figure 83. All other step lap joints were made from the unmachined details that were originally scheduled to be used in the machined step lap program. Three types of multiple-laminate step lap joints were made. The first was a single step lap joint shown in Figure 84 and the second was a double step lap joint shown in Figure 85. Figure 86 presents the triple step lap joint design.

Because of the large overlap involved some volatiles were trapped in the step lap joints, bonded with MB-329 and as a result they had multiple small voids in the bondlines which gave lower than desirable results. The voids were visible, unmagnified on the edges of the bondlines. All AT-120 step lap joints performed were.

\*Registered trade name of the 3M Company

†Registered trade name of the Whittaker Corporation

‡LSA-11, -13, -24 (see Fig. 84)

TABLE XII. ACCEPTANCE TEST\* RESULTS ON AF-126 ADHESIVE (LS-HE)

\*Per MMM-A-132, Type II except 10 specimens instead of 6 were tested. Specimen Average Average Average B-1 B-2 B-4 B-4 B-4 A-1 A-2 ‡No primer used. °Z +Inches. 345/47 345/47 360/40 Batch No. 360 Min. Req. Value.\* 2,500 2.500 2.500 2.500 2,500 2,500 psi \*Per MMM-A-132, Type I, Class 3 except 10 specimens instead of 6 were tested. Date of Previous Results 7/24/70 4/15/70 4/15/79 4/15/70 7/24/70 7/24/70 Previous Results, 5,690 5,772 5,993 5.842 5,092 5,432 psi Cohesive Failure Cohesive Mode Strength, 4,662 4,802 4,845 4,780 Shear 4,421 4,702 5,625 5,627 4,995 5,666 5,160 5,415 5,059 4,818 4,941 5,015 4,995 5,147 4,983 5.266 5.029 5.327 5.117 5.201 5.201 5.092 ‡Probably used wrong primer on these. psi Single Overlap Length† 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 Specimen Average A-4 A-5 Average Average Average Average Average A-4 A-5 B-1 B-2 B-4 B-4 B-5 B-4 A-1 A-2 A-3 A-1 A-2 A-3 No. †Inches. Batch No. 724 739 724 739

2,250

7/24/70

2.574\$

Adhesive Adhesive Adhesive Adhesive

2,972 2,914 2,914 2,826 2,826 2,839

0.500 0.500 0.500 0.500 0.500

Adhesive

Adhesive Adhesive

Adhesive Adhesi /e

Adhesive

2,250

4/15/70

2,653‡ 2,730‡

2,250

4/15/70

Adhesive

Adhesive Adhesive Adhesive

Adhesive

2,250

7/24/70

2.418‡

2,756

J

1."

2,250

7/24/70

2.262‡

TABLE XIII. ACCEPTANCE TEST\* RESULTS ON MB-329 ADHESIVE (HS-LE) Min. Reg.

Date of

Value.\*

Previous Results

Previous Results,

Failure

Mode

Strength,

Single Overlap Length†

psi

Shear

psi

psi

2,250

4/15/70

2,5774

Adhesive

Adhesive Adhesive

Adhesive

Adhesive

2,566 2,488 2,488 2,488 2,488 2,410 2,411 2,628 2,660 2,699 2.717 2.663 2.673 2.674 2.517 2.371 2.371 2.371 2.371 2,468 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 0.500 A-3 A-4 A-5 Average A verage Average A-1 A-2 A-4 A-4 A-5 

Originally, the program's scope covered the manufacture of scarf joints for tests. These were designed to be made as shown in Figure 87. However, a reorientation of the program eliminated their fabrication.

All lap shear assemblies were cut into one-inch wide strips for simple specimen coupon testing.



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FIGURE 80. LAP SHEAR ASSEMBLY SINGLE LAP JOINT

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## FIGURE 81. LAP SHEAR ASSEMBLY - DOUBLE LAP JOINT

### TABLE XIV

State of the state

### SINGLE AND DOUBLE LAP SHEAR ASSEMBLIES - BORON/BORON

L.S.A. No.	Type Adhesive	Adherend No.	Adherend Thickness	Joint Thickness	Bond-Line Thickness	Primer	Corry Pres P	Cure Temp., °F
ı	AF-126	B-21-A	0,044	0.0925	0.0045	Blue EC-2320	50	275
14	AF-126	B-21-F B-21-F B-21-H	0.042	0.091	0.006	Dry at 150° for 30 min		
27	AF-126	B-21-C B-21-J	0.043	0.093	0.007			
2	AF-126	B-24-A B-24-B	0.045	0.0955	0.0035			
15	AF-126	B-28-C B-28-C	0.044	0.0935	0.0075			
28	AF-126	B-24-I) B-24-H	0.046	0.096	0.003			
3	AF-126	B-12-A B-12-D B-13-K	0.016 0.016 0.032	0.065	0.0005			
16	AF-126	B-12-G B-12-1 B-13-F	0,015 0.015 0,032	0.067	0.0025			
29	AF-126	B-1 2-H B-1 2-M B-1 3-M	0.015 0.015 0.032	0.0675	0.00275			
4	AE-126	B-14-J B-14-K B-15-K	0.048 0.046 0.088	0 1845	0.00125			
17	A1-126	B-14-B B-14-F B-15-L	0.046 0.047 0.088	0 184	0.0015			
30	AF-126	B-14-1 B-14-M B-19-K	0.047 0.047 0.087	0.184	0.0015			
5	AF-126	B-24-J B-24-K B-17-A	0.046 0.045 0.091	0.1845	0.00125			
18	AP-126	B-24-F B-24-F B-17-B	0.04 0.046 0.088	0.1845	0.00175			
31	AF-126	B-16-A B-16-G B-22-1	0.047 0.045 0.089	0 1 9 2	0.0055	Blue I C-2320		275
40	MB-329	B-25-4 B-25-J	0.040 0.042	0.086	0.004	Red MB-329 Type If Air dry for 15 min Force dry for 30 min at 235°F		350
53	MB-329	B-254- B-25-K	0,041	0.089	0.007			
66	MB-329	B-25 G B-25 J	0.041	0.0895	0.0075			
41	MB-729	B-28-B B-28-K	0.042	0.095	0.011			
54	MB-329	B-28-D B-28-H	0.045	0.099	0.011			
67	MB-329	B-10 C B-16-F	0.048	0.1025	0.0065			
42	MB-329	B-12-F B-12-K B-13-B	0.016 0.016 0.032	0.0715	0.00375			
55	MB-329	B-12-B B-12-C B-13-J	0.016 0.016 0.032	0.071	0.0035	n e	50	350
68	MB-329	B-12-J B-12-N B-13-L	0.017 0.016 0.033	0,0755	0.00475	Red MB-Bond 329	51	350
43	MB-329	B-14-A B-14-F B-15-J	0.049 0.049 0.088	0.196	0.005			
56	MB-329	B-14-C B-14-G P-19-A	0.047 0.047 0.088	0.192	0.005			
69	MB-329	B-14-D B-14-H B-15-D	0.047 0.049 0.090	0.1935	0.00375			
44	MB-329	B-28-A B-28-E B-17-J	0,046 0,044 0,090	0,1955	0.60775			
57	MB-329	B-24-C B-24-L B-22-K	0.047 0.046 0.087	0,1945	0.00723			
70	мв-329	B-16-B B-16-H B-17-C	0.047 0,047 0.087	0.1945	0.00675		50	350

### TABLE XV

### SINGLE AND DOUBLE LAP SHEAR ASSEMBLIES-BORON/TITANIUM

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			r		r			Cure	Cure	7
1.5	S.A.	Туре	Adherend	Adherend	Joint	Bond-Line Thicknes	Primer	Pressure,	Temp.	•
N	lo.	Adhesive	NO.	INICKIICSS	THICKNESS					-
-	6	AF-126	B-18-C	0.031	0.0655	0.0025	Blue EC-2320	50	275	
1			Ti	0.032	0.0695	0,0085				
	19	AF-120	Ti	0.032	0	0.005			1	
	32	AF-126	B-18-J Ti	0.031 0.032	(),('08	0.005			275	
	45	MB-329	B-18-H	0.031	0.1 655	0.0025	Red M-Bond 329			1
	58	мв-329	T1 B-13-D	0.032	0.070	0.007				
1			Ti	0.032	0.068	0.005			1 250	
	$\frac{\pi}{2}$	MB-329	Ti	0.032	0.0016	0	Blue FC-2320		275	
	7	AF-126	B-16-D	0.047	0.0915		Dide by Done			
	20	AF-126	B-16-E	0.047	0.0955	0.0035				
1	11	AF-126	B-24-M	0.043	0.099	0.007			275	
		MB 120	Ti B-28-F	0.045	0.091	0.003	Red M-Bond 329		350	Ì
	40	MD-327	Ti	0.045	0.001	0.004				1
	59	MB-329	В-28-М	0.044	0.075	0.00		1		
	?2	MB-329	B-16-J	0.048	0.1445	0.0065			350	
	8	AF-126	B-18-A	0.031	0.04		Blue EC-2320			
				0.016	0.064	0.001				
	21	AF-126	B-18-F	0.031	0.063	0				
			Ti	0.016	0.000					
	34	AF-126	B-13-H	0.031	0,0605	0			27	5
			Ti	0.016			Red M-Bond 32 <sup>c</sup>		35	n
	47	MB-329	B-18-D	0,031	0.074	0.055				
		MD.3 10	Ti B-13-C	0.016						
	60	arb-527	Ti	0.016	0.074	0.005				
	73	MB-329	B-18-B	0.018						
			TI	0.016	0.0725	0.0045			35	0
1	9	AF-126	B-15-A	0.090	0.179	0	Blue EC-2320			
				0.045	0.170	0				
1	22	AF-126	B-15-C	0.087	0.178	0.0005				
			Ti	0.045			1			
	35	AF-126	Б-19-В Ті	0.086	0.175	0	Red M-Bond 32	9	2	75
		MD 23		0.045					3	50
	48	MB-32	7 B-13-5 Ti	0.045	0.1875	0.0052	5			
	61	MB-32	9 B-19-J	0.045			1			
	01		Ti	0.045	0.193	0.0065				
	74	MB-32	9 B-15-M	0.087	0.100	0.0046	Red M-Bond 3	29		۷
			Ti Ti	0.045	0.190	0.0000	Pius 5C-1110			50
	10	AF-12	6 B-22-J	0.089	0.180	0.0005	5	V V		
			Ti	0.045			Blue EC-2320	50		275
	23	AF-12	6 B-17-F Ti	0.088	0.181	5 0.001	75			T
			Ti	0.045						
	36	AF-12		0.045	0.180	5 0.000	75			275
	10	MB-3	29 B-22-/	A 0.043	à		Red M-Bond 3	29		350
	"		Ti	0.04	5 0.194	5 0.007	13			
	62	мв-3	29 B-22-	B 0.08	0 107	5 0.009	25			
	1	ļ	Ti Ti	0.04	5	0.007				
	75	MB-3	29 B-17-	L 0.08	9 5 0.195	0.008	3			350
			Ti	0.04	5				·	550





and to the and











## FIGURE 85. MULTIPLE-LAMINATE TWO-STEP I AP JOINT



# FIGURE 86. MULTIPLE-LAMINATE TRIPLE STEP LAP JOINT

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### SECTION IX

### BONDED JOINT TEST RESULTS

### IX.1. GENERAL

This Section presents the experimental results of the simple specimen bonded joint testing conducted in this program. Two hundred and three specimens covering two adhesives, three lap configurations, three basic composite adherend fiber orientations, and two adherend material combinations were based. Six representative samples of these specimens were extensively strain gaged and tested for detailed behavior study.

Section IX.2 presents the Simple Specimen Data Summary based on the detailed data covered in Appendix F. Section IX.3 gives the results of the Special Joint Investigation Data Summary covering the six joints which were extensively strain gaged.

### IX.2. SIMPLE SPECIMEN DATA SUMMARY

A summary of all the simple specimen lap joint tests is contained in Tables XVI through XXIV along with the failed specimen photographs presented in Figures 88 through 103. Each lap shear assembly number identified in these tables represents an average of three or four specimens taken from it and tested. The total number of simple specimens tested was 203 and the detailed data tables are located in Appendix F. There were 72 single lap, 108 double lap, and 23 step lap joints tested covering composite/composite and composite/titanium bonded joints utilizing two adhesive systems: a nitrile-epoxy low stiffness-high elongation (LS-HE) system\* and an epoxy-novolak high stiffness-low elongation (HS-LE) system.† Selectively covered were three basic orientations of the boron/epoxy adherend materials with sequence variations on two of these. The titanium adherend materials were the 6A1-4V alloy in the annealed condition. Data on the adherend materials was previously covered in Section VII.

From Tables XVI and XVIII, the graph in Figure 104 summarizes the single lap composite/composite joint data on the two adhesives whereas the graph in Figure 105 (taken from Tables XVII and XIX) presents the data on the double lap composite/composite joints with the same two adhesives. In Figure 104 on load transfer capability of single lap composite/composite joints it can be noted that the curve slope increases with increased adherend stiffness, decrensed adhesive stiffness, and increased adhesive elongation. By contrast the change in slope due to these same property variables is much less in double lap joints as shown in Figure 105. In fact all other slopes except one are close to being the same. For this exception it appears that the high Poisson's ratio of the basic  $0/\pm 45^\circ$  orientation adherend is detrimental when used with the high stiffness-low elongation epoxy-novolak adhesive. Also presented in Figure 105 are stopes of data points which represent poor quality specimens, illustrating the deleterious effects of interface region failure.

On the composite/titanium joints summarized in Tables XX and XXII the graph in Figure 106 presents the data on the single lap joints with both adhesive systems whereas the graph in Figure 107 (taken from Tables XXI and XXIII) give them for the double lap joints utilizing the two adhesives. The same sort of trend in load transfer capability exhibited in Figure 104 for the single lap ioints is evident in Figure 106. Higher slopes result from higher adherend stiffnesses and adhesive elongations plus lower adhesive moduli. Figure 107 also exhibits the same trends as were shown in Figure 105. In fact all but one of the adherend/adhesive combinations plot on the same line; this one with the  $0/\pm 15^\circ$  orientation adherends bonded together with the high stiffness low elongation epoxy movolak adhesive gives a slightly lower slope, probably because of Poisson's ratio effects. Again, poor quality specimen results are shown. These had poor adhesive materials quality in the bondline. Also the slopes of the double lap joint curves are generally higher than those of the single lap graphs.

\*3M Company's AF-126-2.

†Narmco's Metlbond 329.

### TABLE XVI

### COMPOSITE/COMPOSITE SINGLE LAP JOINT DATA SUMMARY (LS-HE) AF-126-2, Nominal Width--1.000 In.

Assembly	Measured Length Between	Measured Adhesive	Composite Adherend	Composite Adherend Panel	Measured Composite	Measured Overlap	1/1	Faiture Loud,	Adherend Stress at	Adhenive Stress at	Load Transfer at bailure		Fa	llun	: Ty	pe*,	, <b>4</b> .	General Comments
	Lubs, in.	in	Orientation	Number	Thickness, in	in.		lb/in	psi	per	lb/in /ply	ı	2	1	4	5	6	l
ESA I	4-1/8	0.0010	1º lor	B-21A/D	0.047/0.045	0.250	5.556	1,063	23,629	4,247	118	15		7 K	6	'		
15A 14	5-1/4	0.0043	10197	B-211/H	0.0423/0.045	1 250	29 551	5,668	131,928	4,466	630	2		27	52	19	t	
1 SA 27	7-5/8	0.0043	[0].or	B-21C'/J	0 046 3/0.046	1.7357	37 733	7,871	164,928	4,361	875			5	5	23	67:	
1 SA 2	4-1/8	0.0030	[0/→45/0/=45/0] <sub>S</sub>	B-24A/B	8.047/0.047	0 250	5 11 9	1,147	24,945	4,593	127	27		70	,			
ESA IS	31/4	0.0060	0/+45/0/=45/0} <sub>S</sub>	B-2BC/G	0.0433	1 250	28 868	1,938	89,611	3,106	4 3 H	70			26	4		
I SA 28	67/8	0.0042	[0/+45/0/=45/0] <sub>S</sub>	B 24D/H	0.046	2.000	43 47B	4,758	102,770	2,365	529	1		2		.	96**	
•1 -3 adh 2 -7 adh	ester to bor ester to tita	on/epoxy to nium (6 A1	unpeste 4V															
3 5 coh	rsive																	
5 Sinte	rlaminar																	
6 3 othe																		
±Longitudi	inal splittin	r																
(Net secto	on tensioa ,	and longitud	hnal splitting compo	ute														
**Net sect	ion tension	composite																

### TABLE XVII

### COMPOSITE/COMPOSITE DOUBLE LAP JOINT DATA SUMMARY (LS-HE) AF-126-2, Nominal Width = 1.000 In.

Assembly	Measured Length	Measured Adhesive	Composite Adherend	Composite Adherend Papel	Measured Composite Adherend	Measured Overtap	11	Failure Load,	Adherend Stress at Fadure	Adhesive Stress at	Load Transfer at Latlure.		Fa	Jure	1,	pe•	.4	General Comments
NUMBER	Labs, in	in in	Orientation	Number	Thus ness, in	in		lb/m	psi	ры	lb/m ply	1	2	1	4	5	6	
ISA V	4 1/4	0.0023	$2 \times [0]_{M}, [0]_{6T}$	B-12A/D, B-13K	2 × 0.016/ 0.0273	0.250	9   58	1,783	<u>n5,262</u>	3,510	297	50	-	6	3	1		
1 SA 16	4-3/16	0.0048	$2\times \{0\}_{AF}, \{0\}_{BF}$	B-12G/1, B-13F	2 × 0.015/ 0.0313	0.500	16 667	4,005	131,982	3,958	667	18	1	7	5		6011	
1 NA 29	4.5/8	0 (0033	$2 \times [0]_{M}, [0]_{6T}$	В Г2Н/М. В 13М	2 × 0.0157 0.0313	0 750	25.00	4,150	136,807	2,736	692						10011	
154.4	41/8	0.0918	$2 \times [(0/90)_4/0]_F,$ $\{(0/90)_8/0]_F$	H-14J/K, B-15K	2 × 0.0467 0.088	0 21 N	2 477	1,520	17,015	1,439	89	58		12	10			
15417	4 1 3/32	0.0027	$2 \times [(0/90)_4/0]_I$ , $\{(0/90)_8/0\}_I$	B 148/1 . B 151	2 × 0.046/ 0.0867	0.437	5 040	4,782	54,445	5,197	281	21	1	9		١	17••	
LSA 30	4-5/8	0.0009	$2 \times [(0/90)_4/0]_F,$ $\{(0/90)_8/0\}_T$	8-141/M,	2 × 0.047/	0 729	8 179	5,898	66,977	4,090	147						100**11	
18A 5	4 1/4	0.0013	$2 \times [0/+45/0/-45/0]_S,$ $[(0/+45/0/-45)_Q/0]_S$	В 243/К. В-17А	2 x 0.045/ 0.088	0 250	2 841	2,295	25,820	4,523	135	ч	1	4	15			
L SA 18	4 17/32	0.0020	$2 \times [0/+45/0] - 45/0]_S,$ $[(0/+45/0/-45)Q/0]_S$	B-2417/L, B-17B	2 × 0 046/ 0 0877	0 750	8 552	7,018	78,988	4,614	413	18	1	15	25	2		
I SA 31	5-3/16	0.0017	$2 \times [0/+45/0/-45/0]_S,$ $1(0/+45/0/-45)_Q/0]_S$	B-16A/G, B-221	2 × 0.045/ 0.0897	1 250	13 935	8,715	95,621	1,443	513						100**11	

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tNet section tension and longitudinal splitting composite

(Earlure in double adherend partially over and/or adjacent the bond area

\*\*Net section tension - composite

113 affore in single adherend away from the bond area

[ ] Failure in double adherend away from the bond area

### TABLE XVIII

### COMPOSITE/COMPOSITE SINGLE LAP JOINT DATA SUMMARY (HS-LE) MB-329, Nominal Width-1.000 In.

	Vsv. mbls Nucider	Measured Tongth Botween	Measured Adhesive Disckness,	Composite Adherend Liber	Composite Adherend Panet	Measured Composite Adherend	Measured Overlap Length,	1/1	Latture Load, Vb/m	Adherend Stress at Failure,	Adhesive Stress at Failure,	Load Transfer at Failure,	I	Failu	ne 1	l'ype	•,%		General Comments
L		1355,40	in in	Orientation	Number	Thickness, in.	in.			рч	pu	lb/in./ply	1	2	3	4	5	6	
	18840	4-5/16	0.0043	[0] 8 <i>T</i>	B-251 /J	0.0407/0.040	0.250	6.250	1,075	26,348	4,218	134	8		43	49			
	154 53	5-5-16	0 0073	[0] <sub>87</sub>	B-251/K	0 040 3/0 040 3	1.250	31 017	3,045	74,4H 2	2,403	381	28		я	55	9		
	154-66	6	0.0063	$[0]_{RT}$	8-25G/L	0.041/0.0403	2 2 50	55 B31	4,902	120,486	2,160	613			18	70	12		
[	154-41	4 9/16	0.0070	10/+45/0/-45/01s	H-28B/K	0 04 3/0 04 3	0,500	1) 628	880	20,578	1,755	98	86		10	4			
	154.54	5 S/H	0.0117	[0/+45/0/ 45/0] s	B-28D/H	0.044/0.043	1 500	34 884	2,157	49,502	1,418	240	75		7	18			
	1 SA 67	6-1/8	0.0087	$[0/+45/0/-45/0]_S$	B-16C/I	0.047/0.0477	2.500	53 191	3,043	64,311	1,209	3.18	85			12	3		
ŀ	"Ibad																		

### TABLE XIX

### COMPOSITE/COMPOSITE DOUBLE LAP JOINT DATA SUMMARY (HS-LE) MB-329, Notemal Width = 1,000 In.

Assembly	Measured Length	Measured Aphesive Duck pess	Composite Adherend	Composite Adherenc Panel	A tax	Overlap	11	Failure Load,	Adherend Stress at	Adhesive Stress at	Load Transfer		1 4	lu:	e [)	pe•	. 17	Cener +Comments
Turing I	Labs, in	m	Orientation	Numbe:	Die in	in		lh in	ры	ры	Po in pla	1	2	1	•	5	1 2	 
1.5A-4.2	4  /4	0.0034	2 × [0] 37.1967	19-1-21-/K 19-1-318	.* *5/	0.250	7 812	2,165	66,615	4.266	361	27		16	57			
15A 55	4 3 8	0.0055	$2 + [0] \circ_F [0] \circ_F$	в 12Е ( В 13)		0 500	16 667	4,300	132,716	4,248	717	23		5	1		511 :	
1.54-68	47 K	0.0082	2 + 10131.0161	B 121/N B 131	` ∈ 0.016/ ∈ 0 <b>3</b> 0	1.000	11 111	4,012	123,816	1,981	-69	11		7	25	27	1	
1 SA-43	4 1/4	0.0076	$2 \times [(0, 90)_4   0]_T,$ $[(0/90)_5/0]_T$	B-14A B-15J	2 × 0.049/ ±090	0 250	2 7 78	1,718	18,800	3,383	101	17		36	46	1		
1 SA 56	4 172	0.0046	2 ×[(0/90)4/0]7. [(0/90)8-0]7	19-1404G 18-19A	2 × 0.04*/ 0.088	0 750	8 5 2 3	5.048	56,657	3,322	297	15	}	12	52	4	1	
LSA 69	5 5/16	0.0069	$2 \times [(0/90)_4/0]_T$ $[(0/90)_8/0]_I$	B-14D/H B-15D	2 ± 0.047° 0 €883	1.500	16 988	4,907	54,869	1,617	189	56		5	3	18		
15A 44	4 5/16	0.0087	$2 \times \{0/+45/0/-45,\tilde{0}\}_{S}$ $\{(0/+45/0/-45)_{Q}/\tilde{0}\}_{S}$	B-28A/L, B-171	2 × 0 044/ 0 091	0 229	2 602	2,010	22,508	4,423	118	57		10	32	1		
15A 57	47/8	0.0081	$2 \times [0/+45/0] - 45/\overline{0}]_S$ , $[(0/+45/0/-45)_Q/\widetilde{0}]_S$	8-24C/I 8-22K	2 × 0 046/ 0 086	1 000	11 6 28	5 - 1	60.611	2,594	109	35		9	55	1		
1 SA 70	5 1/2	0 (6)65	2 × [0/+45/0/ 45/0]5. [(0/+45/0/ 45) <sub>Q</sub> /0]5	B-15B/H, B-∓7C	2 × 0 047/ 0.088	1 750	19 886	5,073	57.407	1.436	298	58		2	38	2		
* Dud													_				·	

Net section reasion and longitudinal splitting composite

(Enlarge in double adherend partially over and/or adjacent the bond area

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### TABLE XX

### COMPOSITE/TITANIUM SINGLE LAP JOINT DATA SUMMARY (HS-LE) AF-126-2, Nominal Width-1.000 In.

Assembly Number	Measured Length Briween	Measured Adhesive Pikkness,	Composite Adherend Filter	Composite Adherend Panel	Measured Composite Adherend	Measured Titanium Adherend	Measured Overlap Length,	Lh	Failure Load, Ib/in	Adherend Stress at Failure,	Adhenve Stress at Failure,	Load Transfer at Failure, Ibito Joby	_	1	ulur	- I y 4		ч Г.	General Comments
	1755, 10		Onentation	Numiri	Thickness, in.	T DICK DESS, IN .	in					anym ypay			_	_	-		
LSA 6	2 /4	0.0022	10167	10-1.8C	0.030	0.032	0 250	8 13	995	33,166(8)	394K	166	1	15	64	18			
1.SA 1.9	+ 1/16	0.0058	(0) <sub>67</sub>	a irk	0.031	0 032	1 250	40 12 1	4197	133,366(0) 129,258(Tr)	3307	TOKI		13	13	в	12	491	
LSA 12	1.5/8	0.0042	(0) <sub>67</sub>	(3-1 <b>R</b> )	0.0313	0.032	1.647	5 1 89R	4168	133,061 (B) 130,260(Ti)	2470	693						1004	
LSA 7	2 V/H	0.0027	(0/+45/0/-45/0] <sub>8</sub>	B 26D	0 (145	0.045	0 239	5313	1133	25,185(B)	4497	126	55	п	12	2			
LSA 20	31/4	D (K)28	10/+45/0/ 45/01 <i>5</i>	H 161	u ()467	0.045	1 250	27 778	4255	89,979(B) 93,241(Ti)	3351	473		2	3	15	11	67**	
154-13	3 7/H	0.0032	[0/+45/0/_45/ð] <sub>\$</sub>	B-24M	0.0467	0.045	2 060	46 444	50H7	109,325(b) 112,537(1)	2531	565						1991**	
• Ibut																			

tNet section tension and longitudinal splitting composit

Net section tension in stanium

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\*\*Net section lension in composite

General Note-Each assembly represents the average of 3 or 4 tests.

### TABLE XXI

### COMPOSITE/TITANIUM DOUBLE LAP JOINT DATA SUMMARY (LS-HE) AF-126-2, Nominal Width - 1.000 In.

Assembly	Measurd Length	Measured Adheave	Composite Adherend	Composite Adherend	Measured Composite	Measured Estanium	Measured Overlap	1/1	Failur Load,	Adherend Stress at	Adhesive Stress at	l oad Transfer		ı	a ilu	e I j	pr.		General Comments
real of the s	Tabs, in	in	Orientation	Number	Enclaress, in.	Thickness, in.	in		ſb/in	pu pu	pu pu	Po/in /ply	T	2	ţ	1	\$	6	
LSA B	2.5/16	0.0008	101 <sub>67</sub>	B 184	0.029	2 × 0.016	0 239	8 241	2,592	88,943(B) 79.743(Ti)	5,373	432	8	23	47	22			
LSA 21	2 1/2	0.0003	[0]6T	B 181	0.030	2 × 0 016	0.500	16 667	3,497	129,825(B) 121,689(Ti)	1,891	583	2	2	,	7		8211	
t\$A-34	2 5/8	0.0002	[0]6T	B-13H	0.030	2 × 8 816	0 750	25 000	3,351	111,037(B) 104,141(Ti)	2.222	558						100+	
1 SA-9	2 1/4	0.0002	[10/90] <sub>8</sub> /0] <sub>7</sub>	B-15A	0.088	2 × 0 045	0 250	2 841	1,873	70,926(8)	1,681	110	23	18	19	18	2		
1 SA 22	2-1/2	0.0019	[(0/90) <sub>8</sub> /0] <sub>7</sub>	B 150	0.0847	2 × 0.045	0 500	5 90 3	5,220	60,464(B)	5,119	307		8	23	22		34**	
1.5% 35	211/16	0.0009	[(0/90) <sub>8</sub> /0] <sub>F</sub>	B 198	() ()847	2 × 0 043	0 750	8 855	5,930	69,564(B)	1,927	H8						100**	
1 SA-10	2 1/4	0.0010	[(0/+45/0/=45)Q/0]5	B-22J	0.088	2 × 0.045	0 250	2 841	2,368	26,557(B)	4,669	139	7	32	38	21	2		
I SA 23	211/16	0 0030	$[(0)+45/0]-45)Q^{(0)}S$	8-17K	0 (187	2 × 0.045	0.687	7 897	7,450	86,348(B)	5,457	438		8	28	35	29		
t SA 36	3-178	0.0018	10/+45/0/-45)Q/015	8 220	0.089	2 × 0.045	1.250	14 145	9,780	109,111(B) 107,907(Tr)	1,883	575						1001	

Net section tension and longitudinal splitting composite

Pret section tension. Li

\*Net section tension composite

### TABLE XXII

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# COMPOSITE/TITANIUM SINGLE LAP JOINT DATA SUMMARY (HS-LE) MB-329, Nominal Width- 1.000 In.

mment,							
General Co							
	6	Ξ				14	
	\$	•	~	~		18	~
Å,	4	98	Ŧ	2	32	3	42
L al	3	-	15	10	5	æ	1
La Italia	2		32	42	- 1	4	~
	-	Ś	5	~	ŝ		** *
Tranter	ib. in 'phy	130	126	465	801	233	246
Adhenive Strew at	Pier C	3.076	1,548	1.240	066'1	1.438	882
Adherend Stress at	114	25.625(B)	61,079(B)	87.135(8)	22,445(B)	48.(148/B)	46.24 a(B)
Failure Load,	Lb/In		£\$6'I	2,798	\$10	5 (kg ]	2.215
21		8 313	39 432	70 312	11 624	6 I <b>T</b> EE	52 (88.)
Measured Overlap	un un	0.250	1 250	2 250	0.500	1437	2 500
Meatured Titamum	Thickness m	0.012	0 032	0 032	0.045	0.045	0.045
Measured ( ompoute	Thickness, m	0100	1110	0.032	0.043	0.043	0 048
Adherend	Number	влян	H 13D	BIRF	B-261	B-28M	B-16J
Composite Adherend	Orientation	Inler [	[0].T	[6] bT	0/+45/0/ 45/0]5	0/+45/0/ 45/0]S	0/+45/0/ 45/0]S
Measured Adheuve Thick new	n I	0.000	0.0052	0.0048	0.0045	0 0037	0 0107
Measured 1 ength Retween	Tahs, in	2-5/16	5-3/16	4-1/4	3-1/16	\$-7/16	4-9/16
Avermhly Aumber		1-54-45	1.54.58	LSA-71	1.54-46	L.SA-59	LSA-72

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### TABLE XXIII

# COMPOSITE/TITANIUM DOUBLE LAP JOINT DATA SUMMARY (HS-LE) MB-329, Nominal Width--1,000 In.

Assembly Numbr	Measured Length Between	Measured Adhearre Thickness.	Composite Adherend Fiber	Composite Adherend Panel	Measured Composite Adherend	Measured Titanum Adherrod	Measured Overtap	17	Failure Lond.	Adherend Stress at	Adhenve Strea at Fadure	Load Transfer	<u>د</u>	adum.	T yr			General Comments
	Tabs, m.	5	Onentation	Number	Thetness, m.	Thickness, m.			b, 10	2011	, ad	fb/m/ph	-	<u> </u>	-	~	٩	
154.47	2 8/4	0 0059	101 <i>6T</i>	B-18D	0 0297	2 1 0 016	0 250	8 418	2.645	86.6°1+B)	2125	117	5	~	8	**	1	
1.5.4-60	3.3/8	0 0053	[0] 6T	B-1 X	160.0	2 • 0 016	0.500	16129	061.1	42,718(B)	926.1	222	1	- in - n	<u></u>	-		
17 A&I	2-3/4	n 0036	10165	8-188	r c.X03	2 × 0016	000 -	100 10	1.65 1	143 923(B) 136 439(Tu	2,183	732	*		-2	37	33*	
8₽-VSJ	2-1-4	0.0054	1 (0/8(06/0)	B-15B	0 086	2 × 0 045	n 250	:06:	2,508	28.682(B)	4,930	148	2	<u> </u>	1-	8		
1.5.4-61	213/16	0.0067	T   0/8(06/0'-1	8 61	0 0897	2 × 0 045	0.708	7 893	4.7n3	\$2,292(B)	<b>016.</b> [	277	-	80	<u></u>	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~		
1.5A-74	3-7/16	0.0058	1 [0, ¥(06,0)]	B-15M	0.0887	2 × 0 045	1 500	115 41	1.00.4	(\$6,128(B)	1.652	₹.	8	~		8	30.	
151-49	2.5 16	0800.0	110/+45/0/ 451Q/01s	8 22A	L DBU D	2 × 0.045	n 250	2 BUD	1.187	17,956(B)	1,212	66	-	-	7	-		
154-62	-	66(H) U	101-45/01 4510/015	B-22B	1.0.058	2 . 0 045	1 660	11 274	161,	58,268(B)	2.483	SUK		-				
L.S.A 75	31116	0 (#465	101-45-0 4510/6/5	1,19	0 (1 <b>4</b> 63	2 . 0.045	, 64 I	14 919	- 80.	44,02*(B)	2	535	~	4		*		
Piqi.													{	1				
1 Net secto	on tension	÷																
the west	on tension	composite																

TABLE XXIV

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## STEP LAP JOINT DATA SUMMARY (Nominal Width-1.000 In.)

A ssembly Number	No. of Steps	Fabrication of Steps	Length Between Tabs,	Adherend Comb	Adhesive Type	Adhesive Thickness, In	Composite Adherend Fiber	Composite Adherend Panel	Net Composite Adherend Thick page	Net Titanium Adherend	Total Overlap.	1/7	ailure Load.	Adherend Stress at Failure,	Adhesive Stress at Failure,	Load Transfer at Failure,	Ľ.	tilure	Type	94 •	Ger	aetal Comments
			ц.			1	Orientation	Number	IR.	In.	ġ			Isd	nd	lb/in./ply	1 2	m	4	s	9	
LSA-13	-	L	2-1-2	B-T	MB-329	0 0017	$2 \times [0]_{BT}$	B-25C/M	0 086	0.085	2.006	3.600	2.650	(8)054.01	1.325	166	<u>8</u> -	0	55	۰		
LSA-11	-		6-7/16	B-1	MB-329	0.0160	2 × 10 : 45 010 15	B-26E/G	0 1662	0.1762	2.051	2.341	1,745	0,443(B)	847	55	2 22	=	55	-	\$	
LSA-24	-	-	5-7/8	B-T	MB-329	0.0130	$2 \times [0/90_2/0]_{4T}$	B-23J/L	0 1727	0.1767	2.010	1.639	1,492	8.628(B)	741	47	3 47	-	27	11	25†	
LSA-25	7	L	4-7/8	B-T	AF-126-2	0.0010	$3 \times \{0/90_2/0\}_{4T}$	B-23	0 2637	0 265	3.689	3.989	5,403	(0,474(B)	1,462	113	 	1	1	17	t 00	
LSA-26	~		5-11/16	B-T	AF-126-2	0.0058	3 × 1(0 :45/0)Q S	B-26	0 262	0.254	3.879	4.805	11.146	12,401(B)	2,864	232	10		19	44	26‡	
LSA-37	m	-	91/2-9	B-T	AI126-2	0 0059	4 × [0] + F	B-21	0 1797	0 1743	696 1	1.297	11,783	(B)7(B)	5,966	327	10 22	15	ž	1	194	
LSA-12	3	W	4-3/16	B-B	MB-329	0.0070	{0/902/0]4T (0/145/0)2]5	B-23A B-26A	0.0863/ 0.0857	21	0 267	3 116	006	(0,474(B)	3,376	56		31	36	26		
Note: L = M =	Lamina Machin	ting by bondi ing	ing up pref	abricated h	aminates						1	1			]		-	4	]	1		
bid1.																						
+ Net sectu	on tensic	n and longiti	udinal splu	ting-comp	osite																	

‡Net section tension-composite





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FIGURE 93. LSA-57 FAILED SPECIMENS



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FIGURE 95. LSA-19 FAILED SPECIMENS



FIGURE 96. LSA-8 FAILED SPECIMENS







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FIGURE 98. LSA-10 FAILED SPECIMENS

 $P^{\prime}$ 

10 18 18 91



FIGURE 99. LSA-71 FAILED SPECIMENS





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FIGURE 104. SINGLE LAP COMPOSITE/COMPOSITE JOINTS LOAD TRANSFER CAPABILITY







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FIGURE 107. DOUBLE LAP COMPOSITE/TITANIUM JOINTS LOAD TRANSFER CAPABILITY

Limited step lap test data were obtained in this program and are summarized in Figure 108 (taken from Table XXIV). Examination of Figure 108 will reveal that the same trends are in effect as with the single lap joints with the addition of one new trend. This is that the slope also increases as the number of steps are increased from one to three. It should be noted, as shown in Section VIII, that these are single step-lap joints. This configuration is probably more related to single lap joints than the double lap ones, as slope comparisons with Figures 104 and 106 will show. The load transfer shown for these joints is that of the "gross section" rather than the "net section," therefore, this capability looks low for the one step-lap joint. However, net section calculations would double it. Curve 3 does give a higher slope than the best of the single lap data but not quite as good as the best of the double lap data. These trends probably indicate that an even higher number of steps would further increase the slope as would changing the configuration to that of a double step lap. In this case, the low quality specimens had voids in the bondline.

It should be brought to the readers attention that the above graphical analysis was performed only on the composite/composite and composite/utanium joints utilizing the  $0^{\circ}$  and  $0^{\circ}/\pm 45$  general adherend orientations with both adhesives. Data utilizing both adherend combinations and both adhesive selections were also obtained using the  $0/90^{\circ}$  orientation, but not as extensively. Only double lap and step lap data were obtained with this orientation.

Testing of the tensile lap shear simple specimens was performed in a Multirange Bałdwin Test Machine at a constant deflection rate of 0.0025 in./min (over a 2-in. gage length over the bond area).\* An autographically recorded to a detlection curve was obtained to failure on each specimen over this 2-in. gage length with a multi-range TSMD extensioneter. Temperature and humidity during testing were  $70^{\circ} \pm 4^{\circ}$ F and  $50\% \pm 10\%$ , respectively.

Figures 109 and 110 are characteristic of the type of load/deflection<sup>†</sup> curves that were obtained from the joint specimens, the characteristic difference being that the joint with the low stiffness-high elongation adhesive has a curve which exhibits some nonlinearity while the high stiffness-low elongation adhesive curve is linear to failure. The LS-HE adhesive joint is stronger but gives a slightly lower curve slope compared to the HS-LE joint. A detailed description of these two example joints follows:

Figure 109 represents a 1-in, wide double lap joint with 0.045-in, thick 6A1-4V annealed Ti double adherends and a  $[(0/+45/0/-45)Q/0]_S$  17-ply boron epoxy single adherend 0.087 in, thick. The overlap length was 0.687 in, with the AF-126-2 nitrile epoxy low modulus, high elongation adhesive bond. Failure was predominately cohesive and surface resin fracture at a bondline average shear stress of 5,444 psi. Adherend average tensile stress was 86,023 psi. The load transfer capability of joints made with this adhesive system were very high, regardless of adherend stiffness lap length, or lap type.

Figure 110 presents a 1-in. wide double lap joint with 0.047-in. thick  $[0/+45/0/-45/0]_S$  9-ply boron/epoxy double adherends and a 0.087-in. thick  $[(0/+45/0/-45)Q/0]_S$  17-ply boron/epoxy single adherend. Cverlap length was 1.75 in. using the Metlbond-329 high modulus, low elongation adhesive system. Failure was predominately at the interface region between the laminate and the adhesive at a bondline shear stress of 1,154 psi and an adherend average tensile stress of 45,384 psi. The load transfer capability of these joints range from very high to very low depending on adherend stiffness, lap length and type.

By contrast Figure 109 represents a high quality, high performance joint and Figure 110 illustrates a low quality, low performance joint. Their characteristic failures are the key to their quality differences, although the adhesive shear stresses will always be lower with larger overlaps on high modulus, low elongation bondline materials. It has been found that the type of failure typified by the joint whose mechanical behavior curve is shown in Figure 109 has a predictable failing load with the nonlinear bonded joint analysis techniques developed in this program using a maximum stress adhesive failure criterion (i.e., for the low modulus, high elongation adhesive). Failures in the HS-LE adhesive joint can also be predicted by this criterion when failures are cohesive or surface resin fracture

\*i.e., strain rate is 0.00125 in./in./min.

+Deflection divided by the 2-in, gage length gives gross strain.



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FIGURE 108. STEP LAP COMPOSITE/TITANIUM JOINTS LOAD TRANSFER CAPABILITY

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FIGURE 109. LOAD/STRAIN CURVE FOR DOUBLE LAP AF-126-2 BONDED JO<sup>1</sup>NT (Two Inch Gage Length Over Joint Area)



FIGURE 110. LOAD/STRAIN CURVE FOR MB-329 BONDED JOINT (Two Inch Gage Length Over Joint Area)
However, failures cannot be predicted for either adhesive system when failures occur like the one on the specimen whose mechanical behavior is described in Figure 110. This illustrates that a lack of adhesion exists at the interface region and results in behavior which is not predictable by current nonlinear orthotropic mechanics methods without empirical modification. It is suspected that this is a materials and processes quality problem and therefore should not be considered in design/analysis. Whenever such a condition cannot be prevented it should at least be identified and isolated as was done herein in order for the results to be analyzed in a meaningful manner.

The types of failure observed in the simple joint program can be generally divided into the following classes:

- A. Adhesive/Adherend Interface Adhesion Failure
- B. Adherend Surface Resin/Reinforcing Fiber Interface Adhesion Failure
- C. Cohesive Fracture of the Bondline
- D. Cohesive Fracture of the Adherend Surface Resin
- E. Delamination, Splitting
- F. Adherend Net Section Tension

The first two types (A and B) almost always occur as a result of materials or process quality being low. Failures of this type usually occur at lower joint load transfer magnitudes than those failing by other means. The nonlinear bonded joint analytical methods will not predict these failure types or load magnitudes.

Failure modes C and D represent high quality specimens which have predictable failure modes and magnitudes with the nonlinear joint analysis techniques. The maximum stress failure surface can be used to predict the C and D failure types with the nonlinear theory. Occasionally poor adhesive (or matrix resin) material or improper cure will result in low magnitude test values with the C or D failure type.

Delamination and splitting type failure\* is associated with laminate behavior, probably emanating from some sort of micro-mechanical damage occurring during non-failure loading. However, this has not been proven for boron/epoxy laminates. Large differences in modulus, Poisson's ratio, or thermal coefficient of expansion of the two adherend materials could be a contributing factor as well as a low quality laminate. In any case the present nonlinear techniques cannot predict these failure types or magnitudes without further modification.

Adherend net section tension failures<sup>†</sup> were observed on those specimens designed to fail in this fashion and the failure stress magnitudes were approximately equal to the experimentally measured tensile strength of the composite and titanium materials. Failure predictions can be accomplished with the nonlinear techniques using maximum strain failure criteria for the composite adherends and maximum stress failure criteria for the titanium adherends.

### 1X.3. SPECIAL JOINT INVESTIGATION DATA SUMMARY

Six composite/titanium bonded joints were selected from the simple specimen program reported in Section IX.2 for in-depth experimental behavior study. These "special" joints cover the single, double and step-lap configurations with both adhesive systems and one composite material orientation category ( $0 \pm 45^{\circ}$ ). The joints chosen for study were:

No.	Joint Type	Adhesive	No. of Strain Gages
LSA-20-1	S.L.	AF-126-2	4
LSA-23-1	D.L.	AF-126-2	4
LSA-59-1	S.L.	MB-329	2
LSA-62-1	D.L.	MB-329	6
LSA-11-4	I-St.L.	MB-329	5
LSA-26-4	2-St.L.	AF-126-2	6

Detail experimental data at failure on these joints appear in Appendix F.

### \*Type E.

+Type F.

The method used for presenting the experimental data is the familiar stress-strain curve. Stresses used were the average calculated values based on the adherend cross-section area outside the joint overlap. Strains were those measured directly on the adherend surfaces at selected longitudinal centerline locations.

Data on LSA-20-1 are presented in Figure 111. A map of these strains vs overlap location are shown in Figure 112. Strains at four composite adherend stress levels were mapped: 10, 60, 80 and 92.437 (failure) ksi. The first stress level (10 ksi) was picked so that both the average adherend and adhesive behavior were still in the linear tange. The second stress level (60 ksi) was picked so that the average adherend behavior would still be linear (just below the P.L.) but the adhesive would be nonlinear. At the third stress level (80 ksi) both the adherend and the adhesive average stress levels are in the nonlinear range. Failure stress level (92.437 ksi) was selected as the fourth one with fracture occurring by net section tension in the composite at about the same stress as the average tensile strength data on panel B-10 would indicate. Transverse composite adherend strains were high, as would be expected with the high Poisson's ratio of the  $0/\pm 45^\circ$  orientation.

Figure 113 gives the LSA-23-1 joint strain behavior as a function of average adherend stress level. Failure characteristics of this series of joints (covering also LSA-23-2 and 23-3) is shown in Figure 114. Primary failure was caused by cohesive fracture of the adhesive and composite surface resin. Strain maps of both the composite and titanium adherend are shown in Figure 115. Again the four adherend stress levels were picked for the same reasons given above. They are as shown below:

Avg. Adherend	Estimated Adherend	Estimated Bondline
Stress Level (ksi)	Behavior Range	Behavior Range
10	linear	linear
50	linear	nonlinear
80	nonlinear	nonlinear
90.203	failure	failure

The higher transverse strains of the composite adherend can be seen clearly in Figure 115.

Speciman LSA 59-1 strain data are shown in Figure 116 with typical failures shown in Figure 117. The primary mode was failure at the interface of the surface resin and the boron fibers of the composite adherend under the bondline. Results are considered to be low. The strain distribution map is shown in Figure 118. The high transverse strains in the composite adherend can be noted. Three adherend stresses were selected at which to evaluate the strain distribution: 10, 30 and 14.954 (F) ksi. All are in the linear range for the adherend, but the second one could cause nonlinear behavior in the adhesive. The third (failure) adherend stress level selected is definitely such that the adhesive behavior will be in the nonlinear range.

Figure 119 presents the strain data on LSA-62-1 and Figure 120 shows the cohesive fracture of the adhesive as the primary failure mode. Strain map curves covering both the composite and titanium adherends are shown in Figure 121. Since the adherend net section stress vs joint surface strain plots are all linear (with some minor knees evident) only two stress levels were investigated in the strain distribution study: 25 and 56.648 (F) ksi. While the longitudinal (X-dir.) strains in the composites are only slightly higher than those in the titanium at locations just outside the joint, the transverse strains in the composite are substantially higher (than those in the titanium) at all locations.

Strain distribution on joint LSA-11-4 is shown in Figure 122 with a strain map at four adherend stress levels given in Figure 123. Primary failure was by cohesive fracture of the surface resin of the composite with secondary failure at the interface between the adhesive and the composite surface resin. Some cohesive fracture of the adhesive was also evident. Results are considered to be low. All adherend and adhesive stress levels up to failure, which were investigated, are considered to be in the linear behavior range.

Avg. Calculated P/A Stress (ksi) in Composite Adherend Specimen LSA-20-1 Failed at 4400 lb Composite [0/+45/0/-45/0]s (Ref. Page F-4, 0.L. 1,25 014 Appendix F) 104 \*antum 90 80 7.0. 60 50 40 On Composite 30 O Gage #3 O Gage #4 20 10 3000 3500 4000 4500 500 1000 1500 2000 2500 .3-00-3000 -2500 -2000 -1500-1000 -500 0

นปีการแล้วสารประกันที่ไปกรรมปริโป สมัยสารประวัติให้สมัยให้เป็นประทัศรีไป ไม้สุดให้สร้างในการประกันประทัศรีไปปร

loint Strain, μ in./in.



FIGURE 111. SINGLE LAP COMPOSITE/TITANIUM JOINT WITH NITRILE EPOXY ADHESIVE (LS-HE)



FIGURE 112. SINGLE LAP C/T-LSHE JOINT, COMPOSITE ADHEREND STRAIN DISTRIBUTION

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EPOXY ADHESIVE (LS-HE)



FIGURE 114. LSA-23 FAILURE PHOTOGRAPHS





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- **1**'



FIGURE 116. SINGLE LAP COMPOSITE THANICM JOINT WITH EPONY NOVOLAK ADHESIVE (HS-LE)

4 2.07



FIGURE 117. LSA-59 FAILURE PHOTOGRAPHS



FIGURE 118. SINGLE LAP C/T-HSLE JOINT COMPOSITE ADHEREND STRAIN DISTRIBUTION

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 ${\mathcal P}' :$ 



FIGURE 120. LSA-62 FAILURE PHOTOGRAPHS

 $l^{t'}$ 



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FIGURE 121. DOUBLE LAP C'T-HSLE JOINT, COMPOSITE AND TITANIUM ADHEREND STRAIN DISTRIBUTION





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FIGURE 123. ONE-STEP LAP C/T-HSLE JOINT, COMPOSITE ADHEREND STRAIN DISTRIBUTION

r'

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LSA-20-4 specimen strain distribution is given in Figure 124. Failure mode was cohesive fracture of both the adhesive and the surface resin of the composite. Figure 125 shows the strain distribution map for the four adherend stress levels investigated and can be categorized as follows:

Avg. Adherend Stress Level (ksi)	Estimated Adherend Behavior Range	Estimated Adhesive Behavior Range
5	linear	linear
15	linear	nonlinear
30	linear	nonlinear
36.969(F)	linear	nonlinear

The adherend strain map shows that the strain distribution for the two-step-lap joint deviates substantially from that of the single and double lap joints. Maximum adherend surface strains occur in the middle of the joint on this two-step-lap configuration while those on the single and double lap ones are located at the start of the overlap. A sudden change in strain also occurs over the middle riser where the middle layer is changing from composite to titanium. Transverse strains of the composite adherend are relatively high.



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# FIGURE 124. TWO-STEP LAP COMPOSITE/TITANIUM JOINT WITH NITRILE EPOXY ADHESIVE (LS-HE)

Specimen LSA-26-4 (Ref. Page F-5, Appendix F) τ (Not Active)



FIGURE 125. TWO-STEP LAP C/T-LSHE JOINT, COMPOSITE ADHEREND STRAIN DISTRIBUTION





### SECTION X

### THEORETICAL/EXPERIMENTA. BEHAVIOR COMPARISONS AND "EFFECTIVE" PROPERTIES

### X.1. GENERAL

The purpose of this section is to show the correlation between the theoretically predicted joint behavior (including failure loads) and the experimental results on both small simple specimens and larger complex assemblies. Section X.2 covers the Addition of Effective Bending to the theory developed earlier and Section X.3 covers the Analytical/Experimental Behavior Comparison on Simple and Special Joints. Section X.4 presents the Design Analysis of Complex Joints. In Section X.5 Complex Joint Test Data Correlation with Predictive Methods is given.

### X.2. ADDITION OF EFFECTIVE BENDING

The theory outlined in Section II was used to predict the failure loads of several of the experimental joint configurations. Maximum stress theory was used for the adhesive and isotropic adherends. Maximum strain theory was used for the composite adherends. These theories were incorporated into the computer program.

Early investigation indicated that the theory as presented predicted excessively low failure loads. Attempts were made to bring theory into agreement with experiment by an effective properties approach, e.g., modifying the adhesive modulus appropriately. Such attempts were not entirely successful.\*

Comparison of analytical and experimental strains indicated the theory predicted excessively high bending strains. For example, the theory of Section II predicted *compressive* strains in adherend 2 of the single lap at x equal zero,  $y_2$  equal to  $t_2/2$  (refer to Figure 2). This was never observed in the experimental data. Similar inconsistencies arose in the double and step lap. The primary cause of the high predicted bending stresses is the small deflection assumption for the derivation of the equilibrium equations. As the axial load is increased, the eccentricity of the joint is reduced and, therefore, the bending due to the axial load is also reduced. (The reverse occurs in the familiar beam column problem.) Thus, the moment in adherend 2 of the single lap at x equal zero is significantly less than  $P\bar{t}(1 - c/a)/2$  as would be predicted by Equation (9). The plane sections remain plane assumption also exaggerates the bending since in the relativity of the adherend-column for the derivation of an adherend-column for the derivation also exaggerates the bending since in the relativity of the adherend-column for the adherend-column form.

In order to remedy this situation without revising the entire theory, an effective bending factor,  $k_e$ , was introduced. This factor reduced all the computed bending moments in the joint by  $k_e$ . The quantity  $k_e$  was introduced into the equations of Section II in the following form:

Single Lap

$$M_{1} = \frac{1}{2} \left[ \theta + \frac{P\bar{t}}{a} \left( x - \frac{c}{2} \right) - \frac{t_{1}}{2} \phi \right] k_{\theta}$$

$$M_{2} = \frac{1}{2} \left[ \theta - \frac{P\bar{t}}{a} \left( x - \frac{c}{2} \right) + \frac{t_{2}}{2} \phi \right] k_{\theta}$$
(12M)

<sup>\*</sup>While "effective G" could be used in shear distribution and ultimate load prediction, when the normal stresses and the failure theories were introduced, the approach became useless. Large changes (by a factor of 2 to 4) in effective G resulted in only small changes in the predicted failing load.

$$p_{1} = \frac{E}{t(1-\nu^{2})} \left(\frac{1}{D_{1}} + \frac{1}{D_{2}}\right) k_{e}$$

$$p_{2} = \frac{E}{2t(1-\nu^{2})} \left(\frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}}\right) k_{e}$$

$$p_{3} = \frac{G}{t} \left(\frac{1}{A_{1}} + \frac{1}{A_{2}} + \frac{t_{1}^{2}k_{e}}{4D_{1}} + \frac{t_{2}^{2}k_{e}}{4D_{2}}\right)$$

$$p_{4} = \frac{G}{2t} \left(\frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}}\right) k_{e}$$

$$q_{1} = -\frac{PE\bar{t}k_{e}}{at(1-\nu^{2})} \left(\frac{1}{D_{1}} - \frac{1}{D_{2}}\right) \left(x - \frac{c}{2}\right) - \frac{2E}{t(1-\nu^{2})} \left(\frac{M_{1p}}{D_{1}} + \frac{M_{2p}}{D_{2}}\right)$$

$$q_{3} = \frac{PG}{t} \left[\frac{1}{A_{1}} - \frac{1}{A_{2}} + \frac{k_{e}t}{2a} \left(\frac{t_{1}}{D_{1}} + \frac{t_{2}}{D_{2}}\right) \left(\frac{c}{2} - x\right)\right] + \frac{G}{t} \left[\frac{2N_{1p}}{A_{1}} - \frac{2N_{2p}}{A_{2}} - \frac{t_{1}M_{1p}}{D_{1}} + \frac{t_{2}M_{2p}}{D_{2}}\right]$$
(71M)

Double Lap

$$M_1 = \frac{1}{2} \left( \theta - \frac{t_1}{2} \phi \right) k_e \tag{81M}$$

$$p_{1} = \frac{Ek_{e}}{(1 - \nu^{2})tD_{1}}$$

$$p_{2} = \frac{Et_{1}k_{e}}{2(1 - \nu^{2})tD_{1}}$$

$$p_{3} = \frac{G}{t} \left( \frac{1}{A_{1}} + \frac{2}{A_{2}} + \frac{t_{1}^{2}k_{e}}{4D_{1}} \right)$$

$$p_{4} = \frac{Gt_{1}k_{e}}{2tD_{1}}$$
(83M)

Step Lap

$$M_{1} = \frac{1}{2} \left[ \theta + \frac{P}{4} (t_{1} - t_{2}) - \frac{t_{1}}{2} \phi \right] k_{e}$$

$$M_{2} = \frac{1}{2} \left[ \theta - \frac{P}{4} (t_{1} - t_{2}) + \frac{t_{2}}{2} \phi \right] k_{e}$$
(91M)

$$p_{1} = \frac{E}{t(1-\nu^{2})} \left(\frac{1}{D_{1}} + \frac{1}{D_{2}}\right) k_{e}$$

$$p_{2} = \frac{E}{2t(1-\nu^{2})} \left(\frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}}\right) k_{e}$$

$$p_{3} = \frac{G}{t} \left(\frac{1}{A_{1}} + \frac{1}{A_{2}} + \frac{t_{1}^{2}k_{e}}{4D_{1}} + \frac{t_{2}^{2}k_{e}}{4D_{2}}\right)$$

$$p_{4} = \frac{G}{2t} \left(\frac{t_{1}}{D_{1}} - \frac{t_{2}}{D_{2}}\right) k_{e}$$

$$= -\frac{PEk_{e}}{t(1-\nu^{2})} \left(\frac{t_{1}-t_{2}}{4}\right) \left(\frac{1}{D_{1}} - \frac{1}{D_{2}}\right) - \frac{2E}{t(1-\nu^{2})} \left(\frac{M_{1p}}{D_{1}} + \frac{M_{2p}}{D_{2}}\right)$$
(93M)

where the equation numbers of Section II are referred to with the suffix M to indicate Modified. The following values of  $k_e$  were found to give reasonable analytical/experimental correlation:

	Ke
Single Lap	0.01
Double Lap	0.02
Step Lap	0.10

 $q_1$ 

### X.3. ANALYTICAL/EXPERIMENTAL BEHAVIO: COMPARISON ON SIMPLE AND SPECIAL JOINTS

### X.3.a. Simple Specimen Joints

For correlation purposes the averaged results from sixteen lap shear assemblies\* were chosen as representative samples of the 67<sup>+</sup> investigated in this research program. Seven single lap (S.L.), seven double lap (D.L.), and two step lap (St.L.) joints were chosen, covering both adhesive systems and adherend combinations, selectively. Within these categories the choices were made, based on a study of the quality of the experimental data. Bad data points were judged by the failure mode (interface or interlaminar) and/or whether the points fell on or near the majority data test curves (N' vs L/t) as shown in the previous section.

Adherend properties were obtained from Section VII and the literature for use in the computer prediction program. Adhesive properties, taken from the literature, are presented in Table XXV and Figures 126 and 127. A summary of all the properties used in the computer program is given in Table XXVI.

The nonlinear joint behavior equations were programmed for failure by (1) cohesive fracture of the adhesive by the maximum stress theory, (2) tensile failure of the composite adherend by maximum strain theory, and (3) tensile failure of the titanium adherend by maximum stress theory. The experimental/theoretical correlation is presented in Table XXVII. The predicted low values of item 6 may have been caused by the inaccuracy of the unidirectional lamina Ramberg-Osgood three parameter predicted stress-strain curve for the  $0/90^{\circ}$  orientation laminate that was used for all three adherends. Observe that the experimental value in parenthesis correlates reasonably well with that

<sup>\*</sup>Each assembly represents 3 or 4 specimens.

<sup>#</sup>Totaling 203 individual specimens.

### TABLE XXV

### AVERAGE ADHESIVE MECHANICAL PROPERTIES

Domester	Mate	erial
r.openy	AF 126-2 (LSHE)*	MB-329 (HSLE)†
0 <sub>u</sub>	5513 psi	7300 psi
E‡	0.22568 × 10 <sup>6</sup> psi	0.96847 × 10 <sup>6</sup> psi
ν‡	0.40	0.4284
τ <sub>u</sub>	7170 psi	8970 psi
G	0.0806 × 10 <sup>6</sup> psi	0.399 X 10 <sup>6</sup> psi

\*See References (6) and (18).

†See References (18) and (19).

‡These v<sup>-</sup> les are calculated. Such calculations are based on c, perimentally measured G and  $E^*$  (constrained) values assuming an isotropic-elastic relationship among E, G and  $\nu$ . See Reference (19).

predicted. It represents the proportional limit average value for this group of joints and was taken from the 2-in. gage length (over the bondline) autographically recorded load/deflection curves. Item 6 (lap shear assembly 56) load/deflection curves are shown in Figure 128. Apparently, the shape of the three parameter stress-strain curves used to derive the  $0/90^{\circ}$  orientation adherend behavior resulted in adhesive stresses high enough to cause joint failure prediction at the loads indicated. In the actual test adherend proportional limits\* were exceeded and a redistribution of stresses in the adhesive was brought on by the sudden drop in adherend modulus, allowing the bondline to go to a much higher stress, i.e., the joint to transfer much higher loads. It is felt that the proper choice of unidirectional lamina Ramberg-Osgood parameters would give derived  $0/90^{\circ}$  stress-strain curves which would allow accurate prediction of failure loads. Item 13 (lap shear assembly 61) apparently was less affected by this phenomenon because only the single adherend was 0/90 composite with the double adherends being titanium. The limiting factor in these three parameter stress-strain curves' accuracy could be the lack of 90° lamina experimental data.<sup>†</sup>

As pointed out in the previous section, values of the bending factor were selected for reasonable correlation with each joint 'ype (BF-JT) and are listed in Table XXVII. Table XXVIII gives a more explicit listing of the BF-JT factors relating them to the item no3, of Table XXVII.

Figure 129 shows the experimental/analytical correlation with these bending factors by joint type. A limited amount of effort was devoted to further refining the values of the bending factor for various adhesive types and failure types within the joint types (BF-JT, AT, FT). The selected bending factors are shown in Table XXIX and Figure 130 shows the corresponding correlation. This latter approach could be extended to other joint configurations. The refinement of the bending factor definition is limited only by the amount of experimental data available.

Typical failure types are shown photographically in Figures 130 through 135. Figures 131, 132 and 133 show the composite net section tension failure of the LSA-31, -33 and -36 specimens, respectively. Figures 134 and 135 show the cohesive fracture of the composite surface resin and bondline of LSA-56 and -58, respectively. In Figure 136 specimens LSA-61 show primary composite adherend delamination and secondary surface resin fracture.

Five typical computer printouts on the joint predictive techniques are given in Appendix G. These are on items 7 (LSA-20), 9 (LSA-23), and 15 (LSA-26) from Table XXVII. The printouts cover shear (TAU) and normal (SIGMA) stresses at various stations along the joint overlap length for ultimate loads in Appendix G.1. The individual ply stresses at intermediate loads are also presented in Appendix G.2 for items 7 (LSA-20), 14 (LSA-62), and 15 (LSA-26). Correlation of the predicted surface ply stresses on the special test specimens with the experimentally measured strains is given in the next subsection.

\*See Appendix E for appropriate experimental laminate tensile specimen stress-strain curves.

 $\approx$  Duscresearch effort generated only 0° unidirectional lamina axial test data, the 90° axial data were taken from the literature, as were the in-plane shear values.

TABLE XXVI

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## THREE PARAMETER STRESS-STRAIN CURVE VALUES

								ADHEREND	PROPERT	TES							
ltem	Curve	σ <sub>01</sub> , ksi	σ1 <i>i</i> , ksi	$E_i \times 10^{-6}  \mathrm{psi}$	<i>ni</i>	σ <sub>01</sub> , ksi	σ <sub>11</sub> , ksi	$E_i \times 10^{-6}$ psi	n ni	σ <sub>0i</sub> , kst	oli. ksi 1	5 <sub>i</sub> x 10 <sup>-6</sup> psi	ni	00, ksi	a I i, ksi E	, x 10 <sup>-6</sup> ps	1 11
			Composi Panels B-1	ite Adherends 12 (B-12 only)		Panels	Composi B-13, B-18	ite Adherends Avg (B-I 3 thru	u B-19)	Panels E	Compositu 3-21, B-21 /	e Adherends Avg (B-21 thru	B-28)				
	4(05 xs 66)4	I	122.539	25.923	ŀ	1	147.232	27.489	I	I	177.622	28.825					
7	V(1 > SA 30)	I	122.539	127.115	1	1	147.232	1 33.604	=	I	177.622	136.632	1				
ĥ	$T(2 \cdot s \cdot t)$	016.11		2.750	2.541	11.91(		2.750	2.541	11.910		2.750	2.541				
4	(TRI VS YEL)	7.950		0.933	2.991	7.95(	0	0.933	2.991	7.950		0.933	2.991				
			0.01 Ti 6A1-4	16 Sheet 1V Annealed			0.0. Ti 6AI≁	32 Sheet IV Annealed			0.045 Ti 6Al-	Sheet 4V Sheet		1	0.090 S Ti 6AH4V	Sheet / Sheet	
'n	لار AS ( ع SA	137.7	134.5	17.756	38.741	1 29.5	128.2	15.514	75.438	136.6	133.3	17.251	37.246	134.0	130.5	16.096	34.559
			ЧD	HESIVE PRO	PERTIES									1			
		r <sub>0i</sub> , ksi	$\tau_{1i}$ , ksi $G$	$\frac{1}{i} \times 10^{-6}$ psi	ni 10	<u>)</u> , ksi 71	i. ksi $G_i \times$	$10^{-6}$ psi $n_i$									
			AF	е/ Ероху ( LЭНІ 1 26-2		Adnestv	/e-Epoxy/N MB-32	ovolak (HSLE)									
9	(t vs t)L	3.75	3.25	0.0806	6.318	6.60 5	.35 0	.3158 4.22	27								
~	(r vs 7) <sub>C</sub>	3.32		0.1750	2.684			<u></u>									
Subs	cripts:																
A-e L-e: C-c:	xperimental val kperimental val ulculated values	lues obtair ues taken s taken fro	ned in the F from the li m literatur	program (see Se iterature :e	ection VI	G											

### TABLE XXVII

# SUMMARY OF EXPERIMENTAL/THEORETICAL CORRELATION

						General			5	limate Load					
hen	n Joint No	Adhesive	Thickness	{pv	rerend	Adher.	nerimental		Bending Facto	ı - JT*	Bei	ading Factor-J	IT. AT. FT†	Failure	Experimental Failure Description
				-	5	Orient. Code	L Aperimental.	B.I.	Predicted. Ib	Evp./Pred.	B.F.	Predicted. Ib	Exp./Pred.	I ype‡	
	14 (SL)	AF-126-2	0.0043	₿ ©	B ©	[0] <sub>C</sub>	5.668	0.01	6.420	0.883	0.01	6,420	0.883	4.3	Cohesive fracture of composite surface
rı	27 (SL)	AF-126-2	0.0043	B	B 🖰	[0] <sup>c</sup>	7.873	10.0	7.132	1.104	0.01	7.132	1.104	6++,5	resin and bondline Composite adherend tension, longitudinal
~	(JU) 81	AF-126-2	0.0020	B	B (b)	[0/±45] <sub>c</sub>	7,018	0.02	8,486	0.827	0.20	6.986	1.005	3,4	splitting, and delamination Cohesive fracture of bondline and com-
чv,	31 (DL) 53 (SL)	AF-126-2 MB-329	0.0037		@@ #	[0/±45]c	8,715	0.02	9,042	1964	0.02	9,042	0.964	*****	posite surface resin Adherend tension
•	56 (DL)	MB-329	0.0046			3101	870 5		7707	1.101	10.0	77077	-101.1	4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4	Cohesive fracture of composite surface resin and composite/adhesive interface
				(		John of	(3.093)+++		4	++++(\$+6.0)			(0.945)౱	111 110 .+	Conesive iracture of composite surface resin and adherend tension
-	20**** (SL)	AI -126-2	0.0028	e B B	1	[0/±45]c	4.255	0.01	4.677	016.0	10.0	4.677	0.910	6‡‡.4	Adherend tension and surface resin
×	33 (SL)	AF-126-2	0.0032	G B B	Ô	[0/±45]c	5,087	10.0	5,548	216.0	10.0	5.548	0.917	611	cohesive fracture Adherend tension
σ	23**** (DL)	AF-126-2	0.0030	1C	କୁ	[0/±45] <sub>c</sub>	7,450	0.02	8.326	0 895	0.20	7,474	0.997	4, 5, 3	Composite surface resin cohesive frac-
															ture, delamination, cohesive fracture of
	101. 25	C 201 11	0100 0	6	0		Ĩ								bondine
	17(1) 00	7-071-14	0.0018	)		0/:45]	9.7b0		9.430	1037	0.02	9.430	1.037	6++	Adherend tes sion and longitudinal
Ξ	58 (SL)	MB-329	0.0052	в Ф	<u>о</u>	[0] <sup>6</sup>	1,957	0.01	1,446	1.353	10.0	1,446	1.353	4.2.3	splitting Cohesive fracture of composite surface
															resin and bondline and composite/adhosive
12	\$9 (SL)	MB-329	0.0037	B	©	10/= 351,	2.093	10.0	5161	1 300	10 0	1615	1 300	v	interface Cohecine ferrities of commission and con-
				(	(										restn and delamination
13	(1DL)	MB-329	0.0067	13	କୁ	[0/90] c	4,703	0.02	3.458	1.182	0.02	3.958	1 188	5,4	Composite delamination and surface resin
*	62•••• (DL)	MB-329	0.0099	ΠŪ	B	[0/±45]c	(2.850)±+	201	5.792	0.900	0.02	5.792	0.900	7	cohesive fracture Cohesive fracture of composite surface
51	1 3 6	C-961-34	0,0050	0	- -	191 101				f					tesin
	37 (3 St 1)	AF-126-1	6500.0		00	2(0+2/0)	0+1.11	01.0	11.181.11	/ 66 ()		11.151 ####	0.997	5.611	Composite delamination and tension
				)	)	3121				6		t.	800.1	4. 1. 011	Cohesive tracture of surface resin of
															composite, intanium/bondline interface, composite tension, and splitting

 $\cdot P'$ 







### FIGURE 128. JOINT LOAD/DEFLECTION CURVES FOR LSA-56 (D.L.-HSLE)

### TABLE XXVIII

### VARIABLE BENDING FACTOR SELECTIONS: BE/JT (Based on Joint Type Only)

Joint Type	Bending Factor	ltem Numbers, Table XXVII
Single Lap	0.01	1, 2, 5, 7, 8, 11, 12
Double Lap	0.02	3, 4, 6, 9, 10, 13, 14
Step Lap	0.10	15,16



### TABLE XXIX

Joint Type	Adhesive Type	Primary Failure Type	Bending Factor	Item Numbers, Table XXVII
S.L.	LSHE or HSLE	AT or CF	0.01	1, 2, 5, 7, 8, 11, 12
D.L.	LSHE	CF	0.20	3,9
D.L.	LSHE	АТ	0.02	4,10
D.L.	HSLE	CF	0.02	6, 13, 14
2-St.L.	LSHE	ГA	0.10	15
3-St.L.	LSHE	CF	0.01	16
Legend: S.L	. Single Lap	<u></u>		

### BEST BENDING FACTOR SELECTIONS: BF/JT-AT-FT (Based on Joint Type, Adhesive Type, and Failure Type)

Legend: S.L. - Single Lap D.L. - Double Lap St.L. - Step Lap LSHE - Low stiffness/high elongation (AF-126-2) HSLE - High stiffness/low elongation (MB-329) AT - Adherend Tension CF- Cohesive Fracture (Bondline or Composite Surface Resin)



### FIGURE 130. CORRELATION CURVE ON BONDED JOINTS FOR BE/JT-AT-FT



FIGURE 131. LSA-31 FAILURE



FIGURE 132. LSA-33 FAILURE



FIGURE 133. LSA-36 FAILURE

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FIGURE 135. LSA-58 FAILURE



FIGURE 136. LSA-61 FAILURE

### X.3.b. Special Joint Correlations

Figure 137 shows the plots of the predicted surface strains in the overlap area of specimen LSA-20-1 at one load level with the experimentally measured values superimposed. Figure 138 presents the predicted surface strains in the overlap area of specimen LSA-62-1 at one load level with the experimental points superimposed. Correlation is good enough to further verify the nonlinear analysis and design techniques developed herein.

Note that the experimentally measured longitudinal surface strain values correlate very closely to those predicted for both composite and titanium adherends. The experimentally measured transverse strains on the composite are also shown. Since the plane strain assumption is used in the program the predicted transverse strains are zero.

### X.4. DESIGN ANALYSIS OF COMPLEX JOINTS

A more complex\* joint was designed to evaluate the size effects which can be anticipated when designing larger joints. A basic composite/titanium double lap joint configuration five inches wide, was chosen for evaluation. Fight of these joints were designed as shown in Figure 139 (Dwg, No. 03-2587-13), varying overlap length, adhesive, and laminate orientation. Of these, two were chosen for test: the -501 and -509 assemblies. These two joints are analyzed in Table XXX. This empirical analysis considers the possibility of failure in the titanium adherends, the computer adherend, and the boundline. Experimentally determined transmit (Table XI) and composite (Table IX) strength are used along with simple specimen bonded joint (Figs. 107 and 140) test data. Design was based on average test



### FIGURE 137. THEORETICAL/EXPERIMENTAL CORRELATION OF SINGLE LAP JOINT SURFACE STRAINS



### FIGURE 139. COMPLEX JOINT PANELS



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L. A.S.L

### TABLE XXX

### COMFLEX JOINT ANALYSIS

	10	L/t	16.67(Ti) 17.65(B)	27.9(Ti) 29.5(B)										
	6	Unit Load per Ply Strength, N <sub>2</sub> , kips/in./ply	0.944	0.944			<b>I</b>			r				
	s	$2 \times 6 \times 7 =$ Unit Load Strength, $N_2$ , kips/in. (2 adherends)	15.899	15.899			ure Predicted	ace resin) cohesive	ace resin) cohesive					
ron/Epoxy	2	Test <i>FTU</i> , ksi	93.524÷	93.524+			Type Fallu	ine (or surf turv	ine (or surf) ture					
Bo	4	Thick. 7, . in. :h adher.)	0.085	0.085	 		ŕ	Bondl frac	Bondl					
	5	Fiber Drientation (cao	0/±45/0)Q1S	)/+45 (1)g]s	15	loint Data	Avg. Allowable Unit Load for Bondline Strengt V. kips/in.	9.330	6.250					
	+	Panel (	8-27+ [((	B-27÷ [10	4	Simple	ondline rength÷÷ (r, ksi	10	250		(able IN)			
	3	x 3 = t Load ength kips/in.	2.137	2.137	-	.   .	Avg. Bo Shear St at L/	3.1	2		l, aest pas		1.	
nium		Lunit Unit Str	2*		~		ondline Stress <sub>n</sub> . ksi**	)44	i27		perties u		ıp length	
Titaniu	<b>n</b>	Test F <sub>TU</sub> ksi	134.8	134.8		-	Avg. B Shear at Mmi	4.			8-26 pro		y overl:	
		Sheet Thick 12. in.	0.090	060.0	2	3	Avg. Load tips/in.	.304	.304		I B-26, E		livided h	
		Ass'y No.	105	509		int Dat	Test Unit $N_3$ , k (2 adh		18		o Panel		or 12 d	
		Adheswe	LSHI	HSLE	11	Simple Jo	it Avg‡ it Load ly for <i>L/t</i> , ips/in./ply	0.572	).572	a N	identical t	107	est of 3.8	: 140
		Т	5 1.5	5 2.5			Te Un Per P N <sub>3</sub> . k	<u> </u>	÷	[able >	t B-27	igure	small	Figure
		No. Plies	2 × 16	2 × 16			Ass'y No.	- 501	= 509.	* See 1	† Panel	‡See I	* *The	÷ † See

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FIGURE 140. BONDLINE SHEAR STRFSS VS *L*/*t* FOR COMPOSITE/TITANIUM DOUBLE LAP JOINTS

data using no safety factors. Unit failure load is predicted to be cohesive fracture of the bondline or composite surface resin.

Loading of the complex joint was to be done through four 1/2-in. diameter steel bolts at each end which are loaded via two 3/8-in, steel plates. These steel plates are loaded through a clevis by a 2-1/2-in, diameter steel pin (see Fig. 141).\* This setup was mounted in a Baldwin Universal Test Machine for loading. Load introduction analysis follows for the 03-2587-13 Dwg.-501 and -509 complex joints:

\*Dwg. No. 03-2587-14.


FIGURE 141. LOAD INTRODUCTION FIXTURE FOR COMPLEX JOINT

### Sel Assembly

 $V_{a} = 9.330$  lb/m, (from Table XXX)  $V_{b} \propto 8 = 46.650$  lb estimated failure load

 $P = 1.2 \times 46.650 = 69.975$  lb (load intro. des. ld.) Load Boit =  $P/4 = \frac{69.975}{4} = 17.494$  lb/bolt

Since moduli of Ti and  $0/\pm 45^{\circ}$  orientation *B/E* are approximately equal, from Figure 139 let  $T_1 * = 3 \times 0.090 = 0.270$  in. for the single adherend all Ti end and  $T_2 * = 3 \times 0.090 + 2 \times 0.085 = 0.440$  in. for the composite double adherend end.

Using the double adherend end  $T_2$ .

$$C \cdot F_{BR} = \frac{P}{dT_2} = \frac{17.494}{1/2(0.440)} = 79,500 \text{ psi}$$

*TiF<sub>BRU</sub>* = 245,000 psi (Fig. 3.0362, Page 21, Code 3707, AFML-TR-68-115 Vol. II, Jan. 1968)

$$0^{\pm}45 B/E F_{BRU} = 43,000 \text{ psi} (Fig. 6.2.2.20, Design Guide)^{(20)}$$

1.e.,

if the B/E fails in bearing, this amount of load will be picked up by the *Ti* insert and grip plates and all the load would be transferred to the boron through the four bondlines. This then results in the same bearing condition as the single adherend end, i.e.,

$$F_B = \frac{17,494}{1/2(0,270)} = 130,000$$
 psi bearing in *Ti*

i.e., bearing in Ti ok when compared with allowable. Bondline stress then becomes P/AB:

from above 
$$P = 69.975$$
 lb  
 $AB = 4 \times 5 \times 4 = 80$  in.<sup>2</sup>  
 $F_S = \frac{69.975}{80} = 875$  psi

This stress is lower than the lowest value obtained from all the lap joint data, i.e., bond line is ok.

Another check using Figure 107 gives 572 lb/in./ply composite joint allowable (18,304 lb/in.) or 91,520 lb total load allowable.

Therefore load introduction joint is safe with the bond ine being most critical. -509 assembly will not transmit as much loa d, i.e., it is ok.

### **NSCOMPLEX JOINT TEST DATA CORRELATION WITH PREDICTIVE METHODS**

The two complex (double lap) joints, assemblies -501 and -509, of SwRI Dwg, 03-2587-13 (see Fig. 139) were tablicated and tested to further check out the nonlinear theoretical predictive techniques developed in this 1<sup>---</sup> irch effort. These joints represented larger, wider bonded joint structures which can be more easily instrumented for behavior measurement. Twenty-four strain gages were laid on the front and back faces of the joint in the pattern shown in Figure 142. Test setup for the -501 complex joint assembly with the LSHE\* adhesive is shown in Figure 143 whereas Figure 144 shows the -509 assembly with the HSLE† adhesive.

Load rate was 0.00125 in./min over a two-inch gage length in the joint area and strain gage readings were taken automatically at each 1,000 pound increment of load to failure. Load introduction tabs of 6A1.4V annealed titanium were bonded to the specimens, initially with a room temperature setting, two-part epoxy.

Assembly -501 was loaded to 32,200 lb on first loading, at which the load introduction tabs on the bottom became unbonded. New tabs were made, cleaned, and rebonded with AF-126-2 film adhesive. On second loading failure of the tab bond at the top occurred at 48,100 lb. New tabs were made, cleaned and rebonded with AF-126 adhesive. On third loading the failure occurred in the joint by cohesive fracture of the bondline at 56,800 lb (11,340 lb/in.). A summary of the strain gage readings is given in Table XXXI for each of the three loadings.

Assembly -509 was loaded first to 30,675 lb at which failure occurred simultaneously at both ends in the load introduction tab/specimen bondline. New tabs were made, cleaned, and rebonded with AF-126-2 adhesive. The second loading resulted in cohesive fracture of the joint bondline at 33,000 lb (6,590 lb/in.). Table XXXII presents a summary of the strain gage data for each of these loadings.



FIGURE 142. COMPLEX JOINT STRAIN GAGE LOCATIONS

\*LSHE Low Stifness, High Elongation (AF-126-2). \*HSEE High Stiffness, Low Elongation (MB-329). Using the actual measured joint dimensions the nonlinear predictive program was used to predict the failure load and mode of each of the joints. For the -501 joint, failure was predicted at 12,144 lb/in. whereas for the -509 joint, failure was predicted at 5,188 lb/in. Both failure made predictions were for a cohesive fracture of the adhesive. Observed mode of failure was cohesive fracture of bondline in both cases starting near the ends of the composite adherends. Table XXXIII summarizes the measured and predicted values. Complete printouts of these predictions and their related adhesive and adherend stresses are presented in Appendix H. Composite surface strains taken from these predicted failure data are included above the measured strains in Tables XXX and XXXII.

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Observed that the longitudinal strains predicted at gages 1, 3, 5 and 9 (see Fig. 142) on both joints for all loadings correlated well with the measured strains at that load level. Also, observe that the predicted strains at gage 7 do not correlate the measured strains. With gage 7 located near the end of the bondline



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(a) Front



# FIGURE 143. COMPLEX JOINT -501 TEST SET-UP

 $J^{p''}$ 

FIGURE 144. COMPLEX JOINT -509 TEST SET-UP

(b) Back



(a) Front

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# COMPLEX JOINT EXPERIMENTAL DATA WITH HSLE ADHESIVE

### TABLE XXXIII. LARGE JOINT ANALYSIS/TEST COMPARISON

Joint	Standard S Pre	Stress Analysis diction	Nonlin Pre	ear Design diction	Actual E Fr	xperimental ailure
Assembly	$P_u$ , lb/in.	Type Failure	$P_u$ , lb/in.	Type Failure	$P_u$ , lb/in.	Type Failure
-501	9,330	C.F.	12,144	C.F.	11,340*	C.F.
- 509	6,250	C.F.	5,188	C.F.	6,590†	C.F.
*Ret. Tabl	e XXXI		<b></b>		L	
†Ref. Tabl	e XXXII					

overlap on the composite adherend over the area where failure is supposed to occur, this difference in predicted vs actual strains is critical. On -501, note, gage 7 shows a 22% reduction in measured strain from the first loading to the third one, an indication that bondline shear and normal stress peaking has been reduced at this point by the repeated loadings beyond the adhesives' proportional lither the first loading the strain at gage 17 was exactly 50% of that of gage 17 in the corner of the composite adherence the first loading the strain at gage 17 was exactly 50% of that of gage 7, on second loading 17 was 77% of 7, while on third loading gage 17 was reading 169% of gage 7. For -509 a different redistribution phenomena occurred. On first loading the strain reading of gage 17 lagged those of gage 7 only slightly as the loading increased through 24,000 lb and then started dropping as the load was increased to 30,000 lb. Gage 17 strain dropped faster than 7 and it was reading about 10% of 17 at 30,000 lb load. On second loading both 7 and 17 were reading decreasing magnitudes of minus strains to 32,000 lb before they started increasing again.

Plots of the predicted longitudinal strains along the joint overlap length for predicted failure load are shown for -501 in Figure 145 and for -509 in Figure 146. Actual strains at this load level are superimposed on these figures for the last (or failure) loading test sequence and they correlate well with those predicted. Transverse strain predictions are zero due to the plane strain assumption; however, actual transverse strain measurements are shown to be relatively large. This is caused by the large Poisson's atio exhibited by the  $0/\pm 45^\circ$  composite adherends.

A redistribution of the surface ply strains of the composite adherend at or near the ends and corners, apparently resulting from multiple load cycles before failure, is probably caused by a redistribution of the adhesive strains (stresses) at those points. Such a redistribution could cause the test article to fail at higher loads than predicted since peak adhesive stresses (strains) would not be reached as soon. This could be offset by preload damage to the adhersive, thereby reducing its strength. The exact phenomena causing surface strain redistribution and its resulting effects are unknown.

At this point it can be said that there is a substantial amount of correlation of the predicted mechanical behavior with the experimental results on the complex joints with accurately predicted failure loads obtained when using maximum stress theory cohesive fracture of the adhesive. Adherend failure prediction in the joint was not checked experimentally in the complex joints.

Failed specimen photographs for -501 are shown in Figures 147 and 148 whereas Figures 149 and 150 show the -509 failures.



FIGURE 146. C/T COMPLEX JOINT-PREDICTED VS EXPERIMENTAL ADHEREND SURFACE STRAINS. HS-LE ADHESIVE

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### FIGURE 147. COMPLEX JOINT -501 AFTER FAILURE



(a) Front





(a) Front









(a) Front



(b) Back





### SECTION XI

### BONDED JOINT DESIGN CURVES

### XL1. GENERAL

The purpose of this section is to illustrate how the nonlinear design/analysis techniques developed in this program can be used to generate useful design curves on bonded joints. Section X1.2 presents a Discussion of Approach whereas Section X1.3 gives the Design Curves and how to use them.

### XI.2. DISCUSSION OF APPROACH

Design oriented experimental data curves for composite adherend joints can be generated by plotting failure loads vs the geometric parameter L/t. For each composite orientation a plot of running load/ply vs L/t is recommended. The running loads at joint failure in lb/in, are divided by the number of plies to get the running load/ply (N') in lb/in./ply. The L/t parameter may be obtained by dividing the bondline overlap length by the adherend thickness. Always use the smaller of the two adherend thicknesses.

For this project, data from selected, representative joints were picked from Table XXVII for use in design oriented experimental data curve generation. Table XXXIV summarizes the experimental 'analytical data on nine joint groups\* which were used as a basis for design curve prediction along with pertinent, related geometric parameters. These predicted failure load values and the use of N' = 0 when L/t = 0 define these curves (or lines) up to laminate failure. Laminate failure then becomes a cut off at N' =laminate failure load (constant). This cutoff value may be predicted or plugged in from experimental data on laminates. While the cutoff shown here is based on handwitte adherend failure it could just as easily be based on titanium adherend failure (for composite/titanium joints) where this is critical.

### XL3 DESIGNCURVES

The design oriented average experimental data curves generated fall on or very close to the nonlinear analytically predicted joint failure load values as shown in Figure 151. All predicted and actual failure modes correspond except one, that being No. 36. Actual failure of this joint was by laminate adherend tension while its predicted failure was by cohesive fracture of the adhesive<sup>†</sup>. Experimentally measured tensile ultimate strength is used as the horizontal cutoffs for these curves with the points 31, 33, and 36 used for correlation.

Use of such design oriented experimental data curves which allow prediction of average test values for any L/t (or vice-versa) is one method of allowables determination. Statistically based formulas can be applied with such data to obtain reduced values for use as design allowables. Such formulas were developed in Section V on Experimental Design. The 95% confidence design ultimate allowables may be calculated using the general formula (given by equation 172, page — ) in which the average (mean) value can be taken from the Figure 151 curves. The other parameters are known or can be determined from the detail data tabulation of Appendix F.

Once the nonlinear analytical formulas have been checked out with simple lap joint tests utilizing the joint configuration and material combination desired, they can be used to generate a family of curves for design allowables purposes as was done herein. Or the computerized formulas could be used to predict failure loads and types of failure for any specific joint design which could be modeled as single, double, or step lap. If it is desired to use the type

\*Lach joint group is made up of the avg of 3 or 4 test specimens taken from one lap shear assembly.

<sup>+</sup>Changing the bending factor  $k_e$ , from 0.02 to 0.05 predicts adherend tension failure mode but at a somewhat lower tailing for  $k_e$ .

TABLE XXXIV.

BONDED JOINT ANALYTICAL/EXPERIMENTAL DESIGN DAI A SUMMARY FOR 0/±45 COMPOSITE ADHERENDS

Max         Bondline         Average         Bondline         Bondline         Bondline         Bendine         Bendine         Bendine         Failure         Lift           Sters.         515         51         4.671(A.T.)         520         98.81B)         3.537         4.677(A.T.)         520         98.81B)         3.530         5.926(1)         1.737(1)         0.01         A.T.         27.778           562         109.324B)         1.438         1.615(CF.)         180         3.500         5.926(1)         1.737(1)         0.01         A.T.         27.778           3322         112.54(T)         531         5.48(A.T.)         617         119.5(B)         2.750         6.339(2)         1.469(2)         0.01         A.T.         46.444           2322         112.54(T)         7.37(3)         3.29(2)         1.469(2)         0.01         A.T.         46.444           232         93.548         4.610         5.910(4) <th></th> <th>Load Unit Load Insfer, Transfer (ply, Stress, Vin.• Ib/in./ply 255 473 087 562</th> <th></th> <th></th> <th></th> <th></th> <th></th> <th>cdicted</th> <th>Values</th> <th></th> <th></th> <th></th> <th></th>		Load Unit Load Insfer, Transfer (ply, Stress, Vin.• Ib/in./ply 255 473 087 562						cdicted	Values				
47389.38(B) 93.24(T)3.357 $4.677(A.T.)$ 6175.2098.8(B) 102.5(T)3.5905.926(1) 617 $1.737^{(1)}$ 0.01A.T.27.778562109.32(R) 112.54(T)2.5315.548(A.T.)6171195(B) 1230(T)2.7506.339(2) $1.469^{(2)}$ 0.01A.T.27.77823248.05(B)1.4381.615(C.F.)18037.001.1057.477(3)3.921(3)0.01C.F.33.41923248.05(B)3.4439.042'A.T.)53299.2(B)3.705.910 <sup>(4)</sup> 2.214 <sup>(4)</sup> 0.20C.F.8.55251296.62(B)3.4439.042'A.T.)53299.2(B)3.5805.802 <sup>(5)</sup> -961 <sup>(5)</sup> 0.02C.F.8.55251296.62(B)3.4439.042'A.T.)53299.2(B)3.5805.802 <sup>(5)</sup> 2.116 <sup>(4)</sup> 0.20C.F.8.55251396.62(B)3.4439.042'A.T.)53299.2(B)3.5805.802 <sup>(5)</sup> 0.02C.F.7.897513109.1(B)3.8339.430(C.F.)1±553105.2(B)3.7506.501 <sup>(7)</sup> -1.153 <sup>(7)</sup> 0.02C.F.7.4045513109.1(B)3.8339.430(C.F.)1±553105.2(B)3.7505.993 <sup>(6)</sup> 2.156 <sup>(6)</sup> 0.02C.F.7.897513109.1(B)3.8339.430(C.F.)1±553105.2(B)3.7505.993 <sup>(6)</sup> 0.02C.F.13.946513109.1(B)3.8339.430(C.F.)1±5	473         89.30 (B) 93.347(1)         3.357         4.671(A,T,1)         5.20         98.80 (B) 102.3770         1.737(1)         0.01         A.T.         27.778           562         109.324(T) 112.34(T)         2.531         5.548(A,T,1)         617         102.3670         0.01         A.T.         27.778           232         109.324(T)         1438         1,615(C,T)         180         7.30         2.339(2)         1.469(2)         0.01         A.T.         27.778           232         48.06 (B)         1,438         1,615(C,T)         180         7.30         5.39(2)         1.469(2)         0.01         A.T.         27.778           212         98.05 (B)         4,4614         6.986(C,T)         411         78.861         4.500         5.91(4)         2.214(4)         0.20         C.F.         8552           413         78.97(B)         5,433         9.047'A,T.         532         99.3(6)         2.156(6)         0.20         C.F.         7.897           575         109.1(16)         3.883         9.430(C,T).144         555         105.218         3.750         0.51(5)         0.01         C.F.         7.897           575         109.1(16)         3.883         9.430(C,T).144	473 562	Average A Adherend Bc Stress. 1 <i>f_A</i> , ksi <i>r_</i>	verage indline b/in. B. psi	N•. Ib/in./ply	N/ply.	fA ksi	r_B. psi	Max Bondline Shear Stress, 7max, Psif	Corresponding Max Bondline Norma! Stress, o <sub>max</sub> , psi <sup>†</sup>	Bending Factor‡	Actual Failure Type ••	L/1++
562 $109.32(R)$ $2.538(A,T.)$ $617$ $110.25(R)$ $2.730$ $6.339^{(2)}$ $1.469^{(2)}$ $0.01$ $A.T.$ $46.444$ 232 $48.05(R)$ $1.438$ $1.615(C,T.)$ $180$ $37.0$ $1.105$ $7.477^{(3)}$ $3.921^{(3)}$ $0.01$ $C.F.$ $33.419$ 413 $78.9(B)$ $4.614$ $6.986(C,F.)$ $411$ $78.8(B)$ $4.600$ $5.910^{(4)}$ $2.214^{(4)}$ $0.20$ $C.F.$ $33.419$ 512 $96.62(B)$ $3.443$ $9.042^{14}A.T.$ $5322$ $99.2(B)$ $2.990^{2}(6)$ $2.214^{(4)}$ $0.20$ $C.F.$ $33.530$ 512 $96.62(B)$ $3.433$ $9.042^{14}A.T.$ $5332$ $99.2(B)$ $2.940^{14}B$ $2.214^{(4)}B$ $0.20$ $C.F.$ $7.397$ 575 $109.1(B)$ $3.83$ $9.430(C,F.)1t$ $5352$ $5.92^{(6)}B$ $2.166^{(6)}B$ $0.20$ $C.F.$ $7.897$ 575 $109.2(B)$ $3.750$ $5.302^{(6)}B$ $2.166^{(6)}B$	562         103.2000 (12.54Tf)         2.331         5.548(A.T.) (12.54Tf)         617         119.261Tb (13.07Tb)         2.730         6.339(2)         1.469(2)         0.01         A.T         46.444           232         48.054B)         1,438         1,1615(C.F.)         180         37.10         1,105         7,477(3)         3.921(3)         0.01         C.F.         33.419           243         7.451         6.46(C.F.)         411         78.8(B)         4.600         5.910(4)         2.214(4)         0.20         C.F.         35.319           312         9.662(B)         3.443         9.042*A.T.)         532         99.2(B)         3.80         5.90(5)         -961(5)         0.01         C.F.         35.3419           313         9.430(C.F.)1t         532         99.2(B)         5.480         5.991(6)         0.20         C.F.         7.897           305         109.11(B)         3.83         9.430(C.F.)1t         532         105.3(B)         3.750         6.501(7)         -1.153(7)         0.02         C.F.         7.897           305         109.11(B)         3.833         9.430(C.F.)1t         553         105.2(B)         -1.167(8)         0.02         C.F.         13.405	562	89.98(B)	.357	4.677(A.T.)	520	98.8(B)	3,590	5.926(1)	1,737(1)	0.01	A.T.	27.778
232 $1.2.30(1)$ $1.380$ $37.0$ $1.105$ $7.477(3)$ $3.921(3)$ $0.01$ $C.F.$ $33.419$ 413 $7.8.97(B)$ $4.614$ $6.986(C.F.)$ $411$ $78.8(B)$ $4.600$ $5.910^{(4)}$ $2.214^{(4)}$ $0.20$ $C.F.$ $8.552$ 512 $96.62(B)$ $3.443$ $9.042^{*}A.T.$ $532$ $99.2(B)$ $5.90(4)$ $2.214^{(4)}$ $0.20$ $C.F.$ $8.552$ 512 $96.62(B)$ $3.443$ $9.042^{*}A.T.$ $532$ $99.2(B)$ $5.90^{*}(5)$ $-961^{(5)}$ $0.20$ $C.F.$ $8.552$ 512 $96.62(B)$ $3.443$ $9.042^{*}A.T.$ $532$ $99.2(B)$ $5.980^{*}(5)$ $0.02$ $A.T.$ $13.935$ 438 $86.5(B)$ $3.430$ $8.593^{*}(6)$ $2.16^{*}(6)$ $0.20$ $C.F.$ $7.397$ 575 $109.11(B)$ $3.833$ $9.430(C.F.)1\pm$ $535$ $105.2(B)$ $3.750$ $6.501^{*}(7)$ $-1.676^{*}(6)$ $0.02$ $C.F.$ $7.397$ $5732(B)$ $2.42.0(B)$ $2.435^{*}(B)$ $2.5$	232 $112.3011$ $1.438$ $1.615(C, \Gamma.)$ $180$ $1.7.0011$ $1.105$ $7.477(3)$ $3.221(3)$ $0.01$ $C, F.$ $33.419$ 413 $78.96(B)$ $4.614$ $6.986(C, F.)$ $411$ $78.81B$ $4.600$ $5.910^{(4)}$ $2.214^{(4)}$ $0.20$ $C, F.$ $85.32$ 512 $96.62(B)$ $3.443$ $9.047'A, T.$ ) $332$ $9.92(B)$ $3.580$ $5.802(5)$ $-961(5)$ $0.02$ $C, F.$ $8.531$ 512 $96.51B)$ $5.457$ $7.474(C, F.)$ $440$ $86.5(B)$ $3.590^{(6)}$ $2.214^{(4)}$ $0.20$ $C, F.$ $7.897$ 535 $109.11(B)$ $3.833$ $9.430(C, F.)144$ $5.53$ $105.2(B)$ $3.750$ $6.501^{(7)}$ $-1.155^{(6)}$ $0.02$ $C, F.$ $7.897$ $305$ $5827(B)$ $2.381$ $1.1.81(C, F.)$ $341$ $6.501^{(7)}$ $-1.157^{(7)}$ $9.02^{(7)}$ $0.12^{(7)}$ $0.12^{(7)}$ $0.11^{(12)}$ $0.20^{(12)}$ $0.12^{(12)}$ $0.12^{(12)}$ $0.12^{(12)}$ $0.12^{(12)}$ $0.112^{(12)}$ $0.10^{(12)}$ </td <td></td> <td>109.32(B)</td> <td>531</td> <td>5<b>,</b>548(A.T.)</td> <td>617</td> <td>119.5(B)</td> <td>2,750</td> <td>6,339<sup>(2)</sup></td> <td>1,469(2)</td> <td>10.0</td> <td>Α.Γ</td> <td>46.444</td>		109.32(B)	531	5 <b>,</b> 548(A.T.)	617	119.5(B)	2,750	6,339 <sup>(2)</sup>	1,469(2)	10.0	Α.Γ	46.444
413       78.99(B)       4,614       6,986(C.F.)       411       78.8(B)       4,600       5,910 <sup>(4)</sup> 2,214 <sup>(4)</sup> 0.20       C.F.       8.552         512       96.62(B)       3,443       9,042'A.T.)       532       99.2(B)       3,580       5,802 <sup>(5)</sup> $-961^{(5)}$ 0.02       A.T.       13,935         438       86.35(B)       5,457       7,474(C.F.)       440       86.5(B)       5,93(6)       2,156 <sup>(6)</sup> 0.20       C.F.       7,897         575       109,11(B)       3,883       9,430(C.F.)±±       555       105.2(B)       3,750       6,501 <sup>(7)</sup> $-1,153^{(7)}$ 0.02       A.T./L.S       14.045         305       58.27(B)       3,883       9,430(C.F.)±±       555       105.2(B)       3,750       6,501 <sup>(7)</sup> $-1,153^{(7)}$ 0.02       C.F.       7,897         305       58.27(B)       2,483       5,792(C.F.)       341       64.9(B)       2,800       (5.1)55.24(E) $-1,676^{(8)}$ 0.02       C.F       11.274         305       58.27(B)       2,864       11.181(C.F.)       234       42.5(B)       2,800       (5.1)55.24(E) $-3,018(E)$ 0.10       A.T       14.805         2	413       78.94(B) $4,614$ $6,986(C,F,J)$ 411 $78.8(B)$ $4,600$ $5,910^{(4)}$ $2,214^{(4)}$ $0.20$ $C,F$ $85.525$ 512       96.62(B) $3,433$ $9,042^{1}A,T,J$ $532$ $99.2(B)$ $3,580$ $5,802^{(5)}$ $-961^{(5)}$ $0.02$ $A,T$ $13.935$ 438 $86.35(B)$ $5,457$ $7,474(C,F,J)$ $440$ $86.5(B)$ $5,930^{(6)}$ $2,156^{(6)}$ $0.20$ $C,F$ $7397$ 575 $109,11(B)$ $3.833$ $9,430(C,F,J)_{12}$ $341$ $6,43(B)$ $2,196$ $6,201^{(7)}$ $-1,157^{(6)}$ $0.02$ $C,F$ $7397$ 305 $58,37(B)$ $2,483$ $5,792(C,F,J)$ $341$ $6,4,900$ $2,116^{(1)}$ $0.02$ $C,F$ $7,897$ $305$ $105,21(B)$ $2,483$ $5,392(C,F,J)$ $341$ $6,4,900$ $0.02$ $C,F$ $11,244$ $305$ $105,21(B)$ $2,483$ $6,501^{(1)}$ $0,102$ $6,76$ $11,244$ $30,21(F)$ $2,483$ $6,501^{(1)}$ $0,102$ $6,7F$	232	48.05(B)	438	1.615(C.F.)	180	37.0	1,105	7,477(3)	3.921 <sup>(3)</sup>	0.01	C.F.	33.419
512       96.62(B) $3.443$ 9.042'A.T.)       532       99.2(B) $3.580$ $5.802^{(5)}$ $-961^{(5)}$ $0.02$ A.T. $13.935$ 438       86.35(B) $5.457$ $7.474(C.F.)$ 440 $86.5(B)$ $5.93^{(6)}$ $2.156^{(6)}$ $0.20$ $C.F.$ $7.897$ 575 $109,11(B)$ $3.883$ $9.430(C.F.)\pm 1$ $555$ $105.2(B)$ $3.750$ $6.501^{(7)}$ $-1.153^{(7)}$ $0.02$ $A.T./L.S$ $14.045$ $305$ $58.27(B)$ $2.883$ $9.430(C.F.)\pm 1$ $555$ $105.2(B)$ $3.750$ $6.501^{(7)}$ $-1.153^{(7)}$ $0.02$ $A.T./L.S$ $14.045$ $305$ $58.27(B)$ $2.383$ $9.430(C.F.)\pm 1$ $341$ $64.9(B)$ $2.800$ $-1.676^{(8)}$ $0.02$ $C.F$ $11.274$ $305$ $5.792(C.F.)$ $341$ $64.9(B)$ $2.156^{(6)}$ $0.02$ $C.F$ $11.274$ $305$ $5.827(B)$ $2.864$ $11.181(C.F.)$ $234$ $42.5(B)$ $-4.425(M)$ $-4.425(M)$ $-4.425(M)$ $-4.425(M)$ $2.246^{(7)}$	512       96.62(B) $3.443$ 9.047'A.T.)       532       99.2(B) $3.580$ $5.802^{(5)}$ $-961^{(5)}$ $0.02$ A.T.       13935         438       86.35(B) $5.457$ $7.474(CF.)$ 440 $86.5(B)$ $5.936^{(6)}$ $2.156^{(6)}$ $0.02$ A.T.       13935         575 $109.11(B)$ $3.883$ $9.430(CF.)1tt$ $555$ $105.2(B)$ $3.750$ $6.501^{(7)}$ $-1.153^{(7)}$ $0.02$ A.T./LS $14.045$ 305 $5827(B)$ $2.883$ $9.430(CF.)1tt$ $555$ $105.2(B)$ $2.156^{(6)}$ $0.02$ $A.T./LS$ $14.045$ 305 $5827(B)$ $2.883$ $9.430(CF.)1tt$ $555$ $105.2(B)$ $2.176^{(6)}$ $0.02$ $A.T./LS$ $14.045$ 305 $5827(B)$ $2.384$ $42.5(B)$ $2.870$ $8.171(B)$ $-1.676^{(8)}$ $0.02$ $A.T./LS$ $14.045$ $10^{(2)}.91(T)$ $2.484$ $11.181(CF.)$ $2.34$ $42.5(B)$ $-1.676^{(8)}$ $0.10$ $A.T$ $11.246^{(6)}$ $2.864$ $11.1.181(CF.)$ $2.34$ $42.$	413	78.94(B)	,614	6,986(C.F.)	411	78.8(B)	4.600	5,910 <sup>(4)</sup>	2,214(4)	0.20	C.F.	8.552
438       86.35(B)       5,457       7,474(C.F.)       440       86.5(B)       5,480       5,993(6) $2,156(6)$ $0.20$ C.F.       7,897         575       109,11(B)       3,883       9,430(C.F.) $\pm \pm$ 555       105,20(B)       3,750       6,501(7) $-1,153(7)$ $0.02$ C.F.       7,897         305       58.27(B)       2,485       5,792(C.F.)       341       64.9(B)       2,880       8.171(8) $-1,676(8)$ $0.02$ C.F       11.274         205       58.27(B)       2,864       11.181(C.F.)       234       42.5(B)       2,880       (5.15,524(E)) $-4,06(B)$ $-4,06(B)$ $-3,018(E)$ $0.10$ $A.T$ 14.805         2322       42.40(B)       2.864       11.181(C.F.)       234       42.5(B) $-4,25(B)$ $-4,25(B)$ $-4,25(B)$ $-4,25(B)$ $-4,25(B)$ $-4,25(B)$ $-4,26(B)$ <td>438       86.35(B)       5.457       7.474(C.F.)       440       86.5(B)       5.480       5.993(6)       <math>2.156(6)</math> <math>0.20</math>       C.F.       7.897         575       109.11(B)       3.883       9.430(C.F.)<math>\pm</math>       555       105.2(B)       3.750       6.501(7)       <math>-1.153(7)</math>       0.02       A.T./LS       14.045         305       58.27(B)       2.483       5.792(C.F.)       341       64.9(B)       2.870       8.171(8)       <math>-1.676(8)</math>       0.02       C.F       11.274         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B)       2.435(M)       <math>-1.676(8)</math>       0.010       A.T       A.T       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B)       <math>-4.425(M)</math> <math>-1.676(8)</math>       0.010       A.T       I       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B)       <math>-4.425(M)</math> <math>-1.676(8)</math>       0.010       A.T       I       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(M)       <math>-1.676(8)</math>       0.10       A.T       I       14.805         333       5.434.66</td> <td>512</td> <td>96.62(B)</td> <td>.443</td> <td>9,042'A.T.)</td> <td>532</td> <td>99.2(B)</td> <td>3,580</td> <td>5,802(5)</td> <td>-961<sup>(5)</sup></td> <td>0.02</td> <td>A.T.</td> <td>13.935</td>	438       86.35(B)       5.457       7.474(C.F.)       440       86.5(B)       5.480       5.993(6) $2.156(6)$ $0.20$ C.F.       7.897         575       109.11(B)       3.883       9.430(C.F.) $\pm$ 555       105.2(B)       3.750       6.501(7) $-1.153(7)$ 0.02       A.T./LS       14.045         305       58.27(B)       2.483       5.792(C.F.)       341       64.9(B)       2.870       8.171(8) $-1.676(8)$ 0.02       C.F       11.274         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B)       2.435(M) $-1.676(8)$ 0.010       A.T       A.T       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B) $-4.425(M)$ $-1.676(8)$ 0.010       A.T       I       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(B) $-4.425(M)$ $-1.676(8)$ 0.010       A.T       I       14.805         232       42.40(B)       2.864       11.181(C.F.)       234       42.5(M) $-1.676(8)$ 0.10       A.T       I       14.805         333       5.434.66	512	96.62(B)	.443	9,042'A.T.)	532	99.2(B)	3,580	5,802(5)	-961 <sup>(5)</sup>	0.02	A.T.	13.935
575       109,11(B)       3,883       9,430(C.F.)±±       555       105,2(B)       3,750       6,501(7)       -1,153(7)       0.02       A.T./L.S       14,045         305       58.27(B)       2,485       5,792(C.F.)       341       64.9(B)       2,870       8,171(8)       -1,676(8)       0.02       C.F       11,274         305       58.27(B)       2,485       5,792(C.F.)       341       64.9(B)       2,880       (S-1)5,524(E)       +3,018(E)       0.10       A.T./L.S       14.045         232       42.40(B)       2,864       11,181(C.F.)       234       42.5(B)       2,880       (S-1)5,524(E)       +3,018(E)       0.10       A.T       14.805         232       42.40(B)       2,864       11,181(C.F.)       234       42.5(B)       2,880       (S-1)5,524(E)       +3,018(E)       0.10       A.T       14.805         232       42.435(M)       -650(M)       -560(K)       -3,018(E)       0.10       A.T       14.805         233       42.5(B)       2,880       (S-1)5,5,24(E)       +3,018(E)       0.10       A.T       14.805         234       42.5(B)       2,8435(M)       -5,057(K)       -3,018(E)       0.10       A.T       14.805 <td>575       <math>109.11(B)</math> <math>3.883</math> <math>9.430(C.F.)1\pm</math> <math>555</math> <math>105.2(B)</math> <math>3.750</math> <math>6.501(7)</math> <math>-1.153(7)</math> <math>0.02</math> <math>A.T./L.S</math> <math>14.045</math> <math>305</math> <math>58.27(B)</math> <math>2.483</math> <math>5.792(C.F.)</math> <math>341</math> <math>64.9(B)</math> <math>2.870</math> <math>8.171(8)</math> <math>-1.676(8)</math> <math>0.02</math> <math>C.F</math> <math>11.274</math> <math>232</math> <math>42.40(B)</math> <math>2.864</math> <math>11.181(C.F.)</math> <math>234</math> <math>42.5(B)</math> <math>2.880</math> <math>(5.1)5.524(E)</math> <math>+3.08(E)</math> <math>0.02</math> <math>C.F</math> <math>11.274</math> <math>232</math> <math>42.40(B)</math> <math>2.864</math> <math>11.181(C.F.)</math> <math>234</math> <math>42.5(B)</math> <math>2.880</math> <math>(5.1)5.524(E)</math> <math>+3.08(E)</math> <math>0.10</math> <math>A.T</math> <math>14.805</math> <math>232</math> <math>42.40(B)</math> <math>2.864</math> <math>11.181(C.F.)</math> <math>234</math> <math>42.5(B)</math> <math>-1.676(B)</math> <math>0.10</math> <math>A.T</math> <math>14.805</math> <math>332</math> <math>42.436(M)</math> <math>-3.027(E)</math> <math>+3.067(E)</math> <math>-3.067(E)</math> <math>4.1.805</math> <math>881</math> <math>881</math> <math>8.24436(M)</math> <math>-1.6767(E)</math> <math>+3.067(E)</math> <math>-1.4366(M)</math> <math>-1.676(M)</math> <math>-1.676(M)</math> <math>-1.676(M)</math> <math>-1.686(M)</math> <math>-1.686(M)</math> <math>-1.686(M)</math> <math>-1.686(M)</math></td> <td>438</td> <td>86.35(B) 5</td> <td>.457</td> <td>7,474(C.F.)</td> <td>440</td> <td>86.5(B)</td> <td>5.480</td> <td>5,993(6)</td> <td>2,156<sup>(6)</sup></td> <td>0.20</td> <td>C.F.</td> <td>7.897</td>	575 $109.11(B)$ $3.883$ $9.430(C.F.)1\pm$ $555$ $105.2(B)$ $3.750$ $6.501(7)$ $-1.153(7)$ $0.02$ $A.T./L.S$ $14.045$ $305$ $58.27(B)$ $2.483$ $5.792(C.F.)$ $341$ $64.9(B)$ $2.870$ $8.171(8)$ $-1.676(8)$ $0.02$ $C.F$ $11.274$ $232$ $42.40(B)$ $2.864$ $11.181(C.F.)$ $234$ $42.5(B)$ $2.880$ $(5.1)5.524(E)$ $+3.08(E)$ $0.02$ $C.F$ $11.274$ $232$ $42.40(B)$ $2.864$ $11.181(C.F.)$ $234$ $42.5(B)$ $2.880$ $(5.1)5.524(E)$ $+3.08(E)$ $0.10$ $A.T$ $14.805$ $232$ $42.40(B)$ $2.864$ $11.181(C.F.)$ $234$ $42.5(B)$ $-1.676(B)$ $0.10$ $A.T$ $14.805$ $332$ $42.436(M)$ $-3.027(E)$ $+3.067(E)$ $-3.067(E)$ $4.1.805$ $881$ $881$ $8.24436(M)$ $-1.6767(E)$ $+3.067(E)$ $-1.4366(M)$ $-1.676(M)$ $-1.676(M)$ $-1.676(M)$ $-1.686(M)$ $-1.686(M)$ $-1.686(M)$ $-1.686(M)$	438	86.35(B) 5	.457	7,474(C.F.)	440	86.5(B)	5.480	5,993(6)	2,156 <sup>(6)</sup>	0.20	C.F.	7.897
305     58.27(B)     2.485     5.792(C.F.)     341     104.5(B)     2.870     8.171(8)     -1,676(8)     0.02     C.F     11.274       232     42.40(B)     2.864     11.181(C.F.)     234     42.5(B)     2.880     (5-1)5.524(E)     +3.018(E)     0.10     A.T     14.865       232     42.40(B)     2.864     11.181(C.F.)     234     42.5(B)     2.880     (5-1)5.524(E)     -3.018(E)     0.10     A.T     14.865       232     42.4436(M)     -4356(M)     -4556(M)     -4556(M)     -4556(M)     -4556(M)	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	575	109,11(B)	.883	9,430(C.I.)±‡	555	105.2(B)	3,750	6,501 <sup>(7)</sup>	-1,153(7)	0.02	A.T./L.S	14.045
232 42.40(B) 2.864 11.181(C.F.) 234 42.5(B) 2.880 (5-1)5.524(E) +3.018(E) 0.10 A.T 14.865 4.425(M) -458(M) -458(M) -458(M) -458(M) (5-2)4.456(M) -458(M) -458(	232     42.40(B)     2.864     11.181(C.F.)     234     42.5(B)     2.880     (S-1)5.524(E)     +3.018(E)     0.10     A.T     14.805       8     4.425(M)     -435(M)     -435(M)     -435(M)     -435(M)     -435(M)       8     5.432(E)     +3.067(E)     +3.067(E)     14.305     -435(M)     -412(M)	305	58.27(B)	.485	5,792(C.F.)	341	64.9(B)	2.870	8,171(8)	-1,676 <sup>(8)</sup>	0.02	C.F	11.274
		232	42.40(B)	.864	11,18HC.F.)	234	42.5(B)	2,880	(S-1)5,524(E) 4,425(M) (S-2)4,436(M) 5,432(E)	+ 3,018(E) -458(M) -455(M) +3,067(E)	0.10	A.T	14.805

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FIGURE 151. DESIGN CURVES FOR SINGLE, DOUBLE, AND STEP-LAP JOINTS

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of allowable curves generated above directly in design it will be necessary to use a design factor K which would be

$$K = \frac{(DA)}{f_s} = \frac{N'_D}{N'_f}$$
(173)

where

### (DA) = Design allowable value, $N'_D$ lb/in./ply (use eq. 172)

 $f_s$  = Mean value of strength  $N'_f$  lb/in./ply

To use the curve in this fashion the design load in lb/in./ply would be input on the ordinate to curve intersection and the L/t value read on the abscissa at that point. This L/t value would then have to be adjusted as follows:

 $(L/t)_D = \frac{1}{K} (L/t)_f = \frac{N'_f}{N'_D} (L/t)_f$ (174)

where

 $(L/t)_D$  = Design value

$$(L/t)_f$$
 = Mean value read from curve

If the input load is above the horizontal cutoff line (as in Fig. 151) a stronger orientation or composite material must be used.

When using the design predictive formulas or computer programs, the distance between assumed zero bending moment points on the adherend on each side of the joint (quantity a) must be known or very large  $\dagger$  as compared to the overlap length c. That is, the ratio c/a must be known or small  $\dagger$  (in the latter case it can be assumed to be zero).

Design ultimate allowables can be calculated based on the nonlinear design/analysis formulae prediction values. These values are used as mean strength values and can be applied to most any joint design which is or can be broken into single, double, or step lap configurations. If the users then have a large backlog of lap joint test data, typical experimentally based statistical parameters will also be available. These can be used with the predicted mean strength to calculate bonded joint design ultimate allowables for most any adherend/adhesive and configuration combination.

Where insufficient basic adherend or adhesive material properties are known, the use of these formulas will be advantageous. This can be done by using assumed "effective" properties ‡, chosen on a trial and error basis to predict failure loads and correlate them with the results from a few simple lap joint tests. Such a procedure will provide a powerful technique for mean joint strength prediction. However, such "effective" properties may be substantially different from the real ones.

Since the predicted mean strength and type of failure of the complex joints (see Section X.5) is reasonably accurate when compared with experimental results, design allowables calculated from such mean strength predictions should also be accurate. Therefore, the use of the standard 1.5 factor of safety on design limit loads to obtain design ultimate loads should be sufficient to provide ample operational safety for static load conditions at room temperature.

In the form of the Ramberg-Osgood three parameter stress-strain curve values for the adherend orthotropic lamina report propic material and the adhesive.

<sup>\*</sup>This would also be the  $N'_f$  value used in equation 172 to obtain the  $N'_D$  value.

 $<sup>\</sup>pi c$  a should be 1/50 or smaller.

### SECTION XII

### RESULTS, CONCLUSIONS AND RECOMMENDATIONS

### XII.1. GENERAL

The purpose of this section is to provide a brief summary of the results and conclusions which have become evident in the completion of the research and in addition to delineate problem areas which were identified. Recommendations for future research along lines related to this effort but further advanced are also covered. Results and Conclusions are covered in Section XII.2, whereas Section XII.3 covers the Recommendations.

### XII.2. RESULTS AND CONCLUSIONS

In brief, the results of this research have been the development and verification of nonlinear design/analysis techniques for certain types of bonded joints covering static failure in several principal modes at room temperature. The methods developed have proven to be accurate when using basic material behavior characteristics and appropriate empirical bending factors.\* The use of assumed adhesive properties in these formulae can be a reasonably accurate technique as long as some simple joint experimental data are available for use in calculating "effective" properties for comparison. Through the use of appropriate failure criteria for the adherend and adhesive in the nonlinear joint formulae the analytical methods become design-predictive equations which can be used to predict joint mean strength, failure type, and as a basis for average strength curves. Example curves have been generated and their design use explained.

Bonded single, double, and step lap, and scarf joints were studied resulting in nonlinear design/analysis techniques being developed for the first three types of joints, whereas only the differential equations were set up for the scarf joint. Comparative results based on typical joint models were generated by both the theoretical methods and the standard nonlinear discrete element techniques. The latter took considerably more computer time to run than the former one. After the theory was developed to the point where good agreement was obtained with the discrete element method, an experimental effort was initiated to provide final verification of the ponlinear analysis methods.

For the three lap configurations, composite adherends of three fiber orientations with two adhesive systems were utilized in the test program along with two adherend material combinations. A total of 203 simple specimen joints were made and tested along with the necessary characterization tests on the composite and titanium adherend materials. In addition, six of these simple specimens were selected for "special" investigation and were extensively strain gauged in the joint overlap area. Data from these special specimens were correlated with the theoretical behavior prediction methods. The large quantity of simple specimen results allowed the theory to be checked out against many geometric, configuration, and material parameter variations as well as failure mode changes. Finally, two larger, complex joints were designed, built, instrumented, and tested as a final check on the analytical methods. These complex specimens were extensively strain gauged for study of the joint behavior under loading. Experimental verification was successful.

Every effort was made to achieve high quality repeatable processing, inspection, and testing. Existing specifications were utilized as much as possible with new specifications written as required. Basically the philosophy was to (1) rigidly monitor and control the incoming material and its subsequent storage, (2) provide complete traceability records on all materials, processing, and testing, and (3) inspect the fabricated materials and joints as necessary with visual and automatic ultrasonic and radiographic methods. Specimen fabrication and instrumentation was accomplished using the same rigid processing and inspection controls utilized in laminate and joint manufacture. Testing was accomplished in accordance with appropriate specifications with all testing conducted at a constant strain rate and with load and strain data automatically recorded both digitally and with autographic continuous plots. Data reduction and analyses were designed to fit the analytical method verification requirements, causing many details to

\*Necessary because the small deflection assumption was inadequate.

be recorded and analyzed which have not usually been considered important in the past. The detailed traceability records were extremely useful in the data analysis task.

A survey and statistical study of bonded lap joint data in the literature was made early in the program to provide guidance on the experimental effort and insight into the generation of design information.

Problem areas encountered were (1) low and variable boron fiber quality from one prepreg batch to the next, (2) inability to machine steps into boron laminate adherends in preparation for step lap joint fabrication and (3) the difficulty of developing rigorous scarf joint analysis equations. Another small problem was that a closely controlled bondline thickness was not achieved in the experimental effort. Tools for achieving lap shear assemblies with controlled, consistent, and repeatable bondline thicknesses were designed but not fabricated and used because of program economic limitations.

In summary, the above research program accomplished all objectives delineated in Section I. The goal of being able to predict all principal failure modes was only partially achieved, however. Cohesive fracture of the bondline or composite surface resin and adherend net section tension failure are the principal modes predictable by the nonlinear equations developed herein. Interlaminar shear (or longitudinal splitting) failure is not predictable by these methods. Neither is interface (adhesive/adherend or composite surface resin/fiber) failure but it is doubtful that this should be considered an acceptable primary mode of failure since it is related to poor materials and processing quality.

The nonlinear joint analysis techniques developed utilize the Ramberg-Osgood three-parameter stress-strain curves on the adhesive and adherend materials as inputs into program in order that behavior may be predicted throughout the elastic and inelastic range to failure. Maximum stress theory was used for both adhesive and titanium adherend failure prediction, whereas maximum strain theory was used for laminate adherend failure prediction. When these were input the analysis computer programs became design predictive programs for mean strength estimation. At predicted failure load the program prints out the bondline adhesive shear and normal stresses at numerous stations along the overlap length along with individual iamina and isotropic adherend stresses at these points. This provides a complete stress map of the overlap area in digital form. Such information will be useful in joint design analysis as well as post-failure critiques.

### XIL3. RECOMMENDATIONS

The areas which need further study are (1) the adaptation of these nonlinear techniques to predict interlaminar shear (and longitudinal splitting) failure\*. (2) modification of the nonlinear formulas to predict interface failure and correlate it with some materials or processing property which depicts qual? - level, (3) modification of plane strain assumption used in the formulas to predict correct transverse composite adherend lamina strains for various orientations and (4) determination of the small deflection theory's adequacy for nonlinear analysis developed herein. The equations need experimental verification for compressive and combined loadings and for other composite materials which exhibit different behavior patterns such as graphite/epoxy, glass/epoxy, and metal matrix composites. Design analysis application studies need to be made which would check out these formulas against typical airframe component structural joints which have been or could be experimentally evaluated.

A need for experimental study of the detailed joint behavior under repeated loadings<sup>+</sup> is also indicated from the test results of this program. Such loadings, expanded into time and temperature dependent spectrums and or environmental exposures typical of airframe applications would yield much information on the time-temperature dependent and/or environmental effects change of bonded joint behavior under various loadings. It might also be

\*Ref (21) presents methods of predicting interlaminar shear bonded joint failure in the elastic range, however, report was received too late for consideration in present program.

Ref (21) also presents considerable fatigue data on bonded joints but was received too late for consideration herein.

possible to relate these behavior changes to a pattern of changes in "effective" input properties and then use the nonlinear formulae as predictive methods.

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An effort to complete development of the nonlinear analysis equations for the scarf joint is also needed along with the necessary discrete element and experimental checks. This effort would round out the nonlinear methods available to cover all the basic types of load transfer joints used in airframe structures.

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### APPENDICES

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### APPENDIX A

### SCARF JOINT EQUATIONS

### SCARF JOINT EQUATIONS

The scarf joint is idealized as shown in Fig. A. l, the cement thickness being exaggerated for clarity.

We take the coordinate axes, x and z, as shown. The displacement  $\overline{u}_0 = \overline{u}_0(x, z)$  is the x displacement of the right face of the adhesive and  $\overline{u}_1 = \overline{u}_1(x, z)$  is the x displacement of the left face.

The displacements  $u_0$  and  $u_1$  are the x displacements of the centroids of the upper and lower adherends, as shown. The z displacements of the upper and lower adherends are  $w_0$  and  $w_1$  taken positive in the positive z directions. The displacements  $w_0$  and  $w_1$  are the usual "bending deflections" of the adherends.

For the present, we shall assume the material to be elastic-isotropic and consider conditions of plane strain.

Consider the free body of Fig. A.2. Equilibrium of the upper element requires that

$$V_{U} + dV_{U} - V_{U} + \frac{\tau dx}{\cos \frac{\theta}{2}} \sin \frac{\theta}{2} - \frac{\sigma dx}{\cos \frac{\theta}{2}} \cos \frac{\theta}{2} = 0$$

 $^{
m or}$ 

$$\frac{dV_U}{dx} = \sigma - \tau \tan \frac{\theta}{2}$$
(1)

and

$$dN_U - \frac{\tau dx}{\cos \frac{\theta}{2}} \cos \frac{\theta}{2} - \frac{\sigma dx \sin \frac{\theta}{2}}{\cos \frac{\theta}{2}} = 0$$









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$$\frac{\mathrm{dN}_{\mathrm{U}}}{\mathrm{dx}} = \tau + \sigma \tan \frac{\theta}{2}$$
(2)

Similarly, for the lower element we have

$$\frac{dV_{L}}{dx} = \sigma - \tau \tan \frac{\theta}{Z}$$
(3)

$$\frac{\mathrm{dN}_{\mathrm{L}}}{\mathrm{dx}} = -\left(\tau + \sigma \tan \frac{\theta}{2}\right) \tag{4}$$

Equations (1) and (3) give

$$\frac{\mathrm{d}}{\mathrm{dx}} \left( \mathbf{V}_{\mathbf{U}} - \mathbf{V}_{\mathbf{L}} \right) = 0$$

So

$$V_{U} - V_{\underline{r}} = constant$$
 (5)

If we take moments about a transverse section, that is, a section normal to the x axis, we get

$$M_{\rm U} + N_{\rm U} \frac{h_{\rm U}}{2} - M_{\rm L} - N_{\rm L} \frac{h_{\rm L}}{2} - P \frac{(h_{\rm U} - h_{\rm L})}{2} \cos \frac{\theta}{2} = 0$$

 $_{
m or}$ 

$$M_{\rm U} - M_{\rm L} + \frac{(N_{\rm U}h_{\rm U} - N_{\rm L}h_{\rm L})}{2} - 2P_{\rm X}\sin\frac{\theta}{2} = 0$$
 (6)

Considering the deformation of the coment, we have

$$\tau = \frac{G}{t} \cos \frac{\theta}{2} \left[ (\overline{u}_0 - \overline{u}_1) - (v_0 - v_1) \tan \frac{\theta}{2} \right]$$
(7)

$$\sigma = \frac{E_C}{t} \cos \frac{\theta}{2} \left[ \left( \overline{u}_0 - \overline{u}_1 \right) \tan \frac{\theta}{2} + \left( v_0 - v_1 \right) \right]$$
(8)

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where  $G \mbox{ and } E_{C}$  are the shear and tensile moduli of the cement. Noting that

$$h_U = \frac{H}{2} + 2x \tan \frac{\theta}{2}$$
,  $h_L = \frac{H}{2} - 2x \tan \frac{\theta}{2}$ 

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$$\frac{dh_U}{dx} = -\frac{dh_L}{dx} = 2 \tan \frac{\theta}{2}$$
(9)

and

$$u_{0} = u_{0} + \frac{h_{U}}{2} \frac{dw_{0}}{dx}, \qquad \overline{u}_{1} = u_{1} + \frac{h_{L}}{2} \frac{dw_{1}}{dx},$$

$$v_{0} = w_{0}, \qquad v_{1} = -w_{1}$$
(10)

we get

$$\tau = \frac{G}{t} \cos \frac{\theta}{2} \left[ (u_0 - u_1) + \frac{h_U}{2} \frac{dw_0}{dx} - \frac{h_L}{2} \frac{dw_1}{dx} - (w_0 + w_1) \tan \frac{\theta}{2} \right]$$
(11)

$$\sigma = \frac{E_{C}}{t} \cos \frac{\theta}{2} \left[ (u_{0} - u_{1}) \tan \frac{\theta}{2} + \left( \frac{h_{U}}{2} \frac{dw_{0}}{dx} - \frac{h_{L}}{2} \frac{dw_{1}}{dx} \right) \tan \frac{\theta}{2} + (w_{0} + w_{1}) \right]$$
(12)

Differentiation gives

$$\frac{d\tau}{dx} = \frac{G}{t} \cos \frac{\theta}{2} \left[ \frac{d}{dx} (u_0 - u_1) + \frac{1}{2} \left\{ h_U \frac{d^2 w_0}{dx^2} - h_L \frac{d^2 w_1}{dx^2} \right\} \right]$$
(13)  
$$\frac{d\sigma}{dx} = \frac{E_C}{t} \cos \frac{\theta}{2} \left[ \frac{d}{dx} (u_0 - u_1) \tan \frac{\theta}{2} + \frac{1}{2} \tan \frac{\theta}{2} \left\{ h_U \frac{d^2 w_0}{dx^2} - h_L \frac{d^2 w_1}{dx^2} \right\} + \sec^2 \frac{\theta}{2} \frac{d}{dx} (w_0 + w_1) \right]$$
(14)

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We assume the following strain-displacement and moment curvature relations for the adherend

$$N_{U} = \frac{Eh_{U}}{1 - v^2} \frac{du_{0}}{dx}$$
(15)

$$N_{L} = \frac{Eh_{L}}{1 - v^{2}} \frac{du_{l}}{dx}$$
(16)

$$M_{U} = -\frac{Eh_{U}^{3}}{12(1-v^{2})} \frac{d^{2}w_{0}}{dx^{2}}, \qquad M_{L} = -\frac{Eh_{L}^{3}}{12(1-v^{2})} \frac{d^{2}w_{1}}{dx^{2}}$$
(17)

where E is the elastic modulus of the adherend and v is Poisson's ratio.

From Equations (15), (16), and (17), we get

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$$\frac{du_{0}}{dx} - \frac{du_{1}}{dx} = \frac{(1 - \nu^{2})}{E} \left( \frac{N_{U}}{h_{U}} - \frac{N_{L}}{h_{L}} \right)$$

$$h_{U} \frac{d^{2}w_{0}}{dx^{2}} - h_{L} \frac{d^{2}w_{1}}{dx^{2}} = -\frac{12(1 - \nu^{2})}{E} \left( \frac{M_{U}}{h_{U}^{2}} - \frac{M_{L}}{h_{L}^{2}} \right)$$

$$\frac{d^{2}w_{0}}{dx^{2}} + \frac{d^{2}w_{1}}{dx^{2}} = -\frac{12(1 - \nu^{2})}{E} \left( \frac{M_{U}}{h_{U}^{3}} + \frac{M_{L}}{h_{L}^{3}} \right)$$
(18)

Substitution into Equations (13) and (14) gives

$$\frac{d\tau}{dx} = \frac{(1 - \nu^2)G}{Et} \cos \frac{\theta}{2} \left[ \left( \frac{N_U}{h_U} - \frac{N_L}{h_L} \right) - 6 \left( \frac{M_U}{h_U^2} - \frac{M_L}{h_L^2} \right) \right]$$

$$\frac{d\sigma}{dx} = \frac{(1 - \nu^2)E_C}{Et} \sin \frac{\theta}{2} \left[ \left( \frac{N_U}{h_U} - \frac{N_L}{h_L} \right) - 6 \left( \frac{M_U}{h_U^2} - \frac{M_L}{h_L^2} \right) + \frac{2E \operatorname{cosec} \theta}{(1 - \nu^2)} \frac{d}{dx} (w_0 + w_1) \right]$$
(19)

Differentiation of Equation (20) and substitution from the third of Equations (18) gives

$$\frac{d^{2}\sigma}{dx^{2}} = \frac{(1 - \nu^{2})E_{C}}{Et} \sin \theta \left[ \frac{d}{dx} \left( \frac{N_{U}}{h_{U}} - \frac{N_{L}}{h_{L}} \right) - 6 \frac{d}{dx} \left( \frac{M_{U}}{h_{U}^{2}} - \frac{M_{L}}{h_{L}^{2}} \right) - 6 \frac{d}{dx} \left( \frac{M_{U}}{h_{U}^{3}} + \frac{M_{L}}{h_{L}^{3}} \right) \right]$$
(21)

From Equations (1) and (3), we get

$$\sigma - \tau \tan \frac{\theta}{2} = \frac{1}{2} \frac{d}{dx} (V_U + V_L)$$
(22)

and, from Equations (2) and (4), there results

$$\tau + \sigma \tan \frac{\theta}{2} = \frac{1}{2} \frac{d}{dx} (N_{U} - N_{L})$$
(23)

Solving Equations (22) and (23) for  $\sigma$  and  $\tau$  , there results

$$\tau \sec^2 \frac{\theta}{2} = \frac{1}{2} \left[ \frac{d}{dx} \left( V_U + V_L \right) + \tan \frac{\theta}{2} \frac{d}{dx} \left( N_U - N_L \right) \right]$$
(24)

$$\tau \sec^2 \frac{\theta}{2} = \frac{1}{2} \left[ \frac{d}{dx} \left( N_U - N_L \right) - \tan \frac{\theta}{2} \frac{d}{dx} \left( V_U + V_L \right) \right]$$
(25)

Since

$$V_{\rm U} = \frac{\mathrm{d}M_{\rm U}}{\mathrm{d}x}$$
,  $V_{\rm L} = \frac{\mathrm{d}M_{\rm L}}{\mathrm{d}x}$ 

these equations may be written in the form

$$\sigma = \frac{1}{2}\cos^2\frac{\theta}{2}\left[\frac{d^2}{dx^2}\left(M_U + M_L\right) + \tan\frac{\theta}{2}\frac{d}{dx}\left(N_U - N_L\right)\right]$$
(26)

$$\tau = \frac{1}{2} \cos^2 \frac{\theta}{2} \left[ \frac{d}{dx} (N_U - N_L) - \tan \frac{\theta}{2} \frac{d^2}{dx^2} (M_U + M_L) \right]$$
(27)

We note the following identities:

$$\frac{N_{U}}{h_{U}} - \frac{N_{L}}{h_{L}} = \left(\frac{1}{h_{U}} - \frac{1}{h_{L}}\right)(N_{U} + N_{L}) + (N_{U} - N_{L})\left(\frac{1}{h_{U}} + \frac{1}{h_{L}}\right)$$
(28)

$$\frac{M_{\rm U}}{h_{\rm U}^2} - \frac{M_{\rm L}}{h_{\rm L}^2} = \left(\frac{1}{h_{\rm U}^2} - \frac{1}{h_{\rm L}^2}\right) (M_{\rm U} + M_{\rm L}) + (M_{\rm U} - M_{\rm L}) \left(\frac{1}{h_{\rm U}^2} + \frac{1}{h_{\rm L}^2}\right)$$
(29)

Since

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$$N_{\rm U} + N_{\rm L} = P \cos \frac{\theta}{2} \tag{30}$$

and, from Equation (6),

$$M_{U} - M_{L} = 2P_{x} \sin \frac{\theta}{2} + \frac{1}{4} [(N_{U} + N_{L})(h_{U} + h_{L}) + (N_{U} - N_{L})(h_{U} + h_{L})]$$
$$= 3P_{x} \sin \frac{\theta}{2} + (N_{U} - N_{L})\frac{H}{4}$$
(31)

Equations (28) and (29) give

$$\frac{N_{U}}{h_{U}} - \frac{N_{L}}{h_{L}} = \left[ -4Px \sin \frac{\theta}{2} + (N_{U} - N_{L})H \right] \frac{1}{h_{U}h_{L}}$$

$$\frac{M_{U}}{h_{U}^{2}} - \frac{M_{L}}{h_{L}^{2}} = \frac{1}{h_{U}^{2}h_{L}^{2}} \left\{ -4x \tan \frac{\theta}{2} \operatorname{el}(M_{U} + M_{L}) + \left[ 3Px \sin \frac{\theta}{2} + N_{U} - N_{L} \right] \frac{H}{4} \right]$$

$$\times (h_{U}^{2} + h_{L}^{2}) \left\{ \right\}$$

Substitution of these results in Equations (19) and (21) gives

$$\frac{d\tau}{dx} = \frac{(1 - \nu^2)G}{Et h_U h_L} \cos \frac{\theta}{2} \left[ -4Px \sin \frac{\theta}{2} + H(N_U - N_L) - \frac{6}{h_U h_L} \left\{ -4Hx \tan \frac{\theta}{2} (M_U + M_L) + 3Px(h_U^2 + h_{L}^2) \sin \frac{\theta}{2} + \frac{H}{4} (h_U^2 + h_L^2)(N_U - N_L) \right\} \right]$$

$$= \frac{(1 - \nu^{2})G}{Et h_{U}h_{L}} \cos \frac{\theta}{2} \left\{ - \left[ 2 + \frac{9(h_{U}^{2} + h_{L}^{2})}{h_{U}h_{L}} \right] P_{X} \sin \frac{\theta}{2} + \left[ 1 - \frac{3}{2} \frac{(h_{U}^{2} + h_{L}^{2})}{h_{U}h_{L}} \right] H(N_{U} - N_{L}) + \frac{24H_{X}}{h_{U}h_{L}} \tan \frac{\theta}{2} (M_{U} + M_{L}) \right\}$$
(32)  
$$\frac{d^{2}\sigma}{dx^{2}} = \frac{(1 - \nu^{2})E_{C}}{Et} \sin \frac{\theta}{2} \left[ \frac{d}{dx} \left\{ -\frac{4P_{X}}{h_{U}h_{L}} \sin \frac{\theta}{2} + \frac{H}{h_{U}h_{L}} (N_{U} - N_{L}) \right\} \right]$$
$$- 6 \frac{d}{dx} \left\{ -\frac{4xH}{h_{U}^{2}h_{L}^{2}} \tan \frac{\theta}{2} (N_{U} + M_{L}) + \frac{(h_{U}^{2} + h_{L}^{2})}{h_{U}^{2}h_{L}^{2}} \left( 3P_{X} \sin \frac{\theta}{2} + (N_{U} - N_{L}) \frac{H}{4} \right) \right\} - 3 \operatorname{cosec} \theta \left\{ \left( \frac{1}{h_{U}^{3}} + \frac{1}{h_{L}^{3}} \right) (M_{U} + M_{L}) + \left( \frac{1}{h_{U}^{3}} - \frac{1}{h_{L}^{3}} \right) \left[ 3P_{X} \sin \frac{\theta}{2} + \frac{H}{4} (N_{U} - N_{L}) \right] \right\}$$
(33)

Differentiation of Equations (26) and (27) and introduction into Equations (32) and (33) yield

$$\frac{1}{2}\cos^{2}\frac{\theta}{2}\left[\frac{d^{2}}{dx^{2}}(N_{U} - N_{L}) - \tan\frac{\theta}{2}\frac{d^{3}}{dx^{3}}(M_{U} + M_{L})\right]$$

$$= \frac{(1 - \nu^{2})G}{Et h_{U}h_{L}}\cos\frac{\theta}{2}\left\{-\left[2 + \frac{9(h_{U}^{2}h_{L}^{2})}{h_{U}h_{L}}\right]P_{X}\sin\frac{\theta}{2} + \left[1 - \frac{3}{2}\frac{(h_{U}^{2} + h_{L}^{2})}{h_{U}h_{L}}\right]H(N_{U} - N_{L}) + \frac{24H_{X}}{h_{U}h_{L}}\tan\frac{\theta}{2}(M_{U} + M_{L})\right\}$$
(34)

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$$\frac{1}{2}\cos^{2}\frac{\theta}{2}\left[\frac{d^{4}}{dx^{4}}\left(M_{U}+M_{L}\right)+\tan\frac{\theta}{2}\frac{d^{3}}{dx^{3}}\left(N_{U}-N_{L}\right)\right]$$

$$=\frac{(1-\nu^{2})E_{C}}{Et}\sin\frac{\theta}{2}\left\{\frac{d}{dx}\left[-\frac{4Px}{h_{U}h_{L}}\sin\frac{\theta}{2}+\frac{H}{h_{U}h_{L}}\left(N_{U}-N_{L}\right)\right]\right]$$

$$=6\frac{d}{dx}\left[-\frac{4xH}{h_{U}^{2}h_{L}^{2}}\tan\frac{\theta}{2}\left(M_{U}+M_{L}\right)+\frac{(h_{U}^{2}+h_{L}^{2})}{h_{U}^{2}h_{L}^{2}}\left(3Px\sin\frac{\theta}{2}+\frac{H}{h_{U}}\left(M_{U}+M_{L}\right)+\frac{(h_{U}^{2}+h_{L}^{2})}{h_{U}^{2}h_{L}^{2}}\left(3Px\sin\frac{\theta}{2}+\frac{H}{h_{U}}\left(M_{U}+M_{L}\right)+\frac{(\frac{1}{h_{U}^{3}}-\frac{1}{h_{L}^{3}}\right)\left(M_{U}+M_{L}\right)+\frac{(\frac{1}{h_{U}^{3}}-\frac{1}{h_{L}^{3}}\right)\left\{3Px\sin\frac{\theta}{2}+\frac{H}{4}\left(N_{U}-N_{L}\right)\right\}\right]\right\}$$
(35)

Equations (34) and (35) represent the two basic differential equations in  $(M_U + M_L)$  and  $(N_U - N_L)$  that are to be integrated. The coefficients in the equations are variable, being functions of the coordinate x. The integral of these equations is not known.

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### APPENDIX B

### STANDARD CONSTITUENT PROPERTIES OF BORON-EPOXY LAMINATES\*

\*Taken from 1st Edition Structural Design Guide for Advanced Composite Applications, August 1969.

### TABLE B.1. BORON FIBER PROPERTIES (RT)

Property	Value
Diameter	0.004275 in.
Density	0.100 lb/in. <sup>3</sup>
a	2.7 $ imes$ 10 <sup>-6</sup> in./in./°F
ν	0.20
$F^{tu}$	450.0 ksi
Et	58.0 $\times$ 10 <sup>6</sup> psi
G	24.2 $\times$ 10 <sup>6</sup> psi

# TABLE B.2. 104 GLASS SCRIM/5505 LAMINATE PROPERTIES (RT)

Property	Value
Density	
аŢ	$9.5 \times 10^{-6}$ in./in./°F
a L	5.8 × 10 <sup>-6</sup> in./in./°F
${ m F}_{ m L}^{ m tu}$	37.8 ksi
$F_{L}^{tpl}$	33.3 ksi
${}^{\rm E}{}^{ m t}_{ m L}$	$3.2 \times 10^{6} \text{ psi}$
$^{vt}_{LT}$	0.151 (at 5000 $\mu$ -in./in. strain)
${}^{\mathrm{ftu}}_{\mathrm{T}}$	13.41 ksi
$_{\mathrm{T}}^{\mathrm{tpl}}$	9.00 ksi
${\mathbb E}^{t}_{\mathrm{T}}$	1.7 × 10 <sup>6</sup> psi
vt TL	0.120 (at 5000 µ-in./in. strain)
Fcu L	45.36 ksi
$_{\rm F_L^{cp\ell}}$	8.88 ksi
$E^{c}_{L}$	4.69 $\times$ 10 <sup>6</sup> psi
v <sup>C</sup> LT	0.32 (at 5000 µ-in./in. strain)
F <sup>su</sup>	11.11 ksi
Fspl	2.10 ksi
G	$0.933 \times 10^6 \text{ psi}$

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## TABLE B. 3. 2387 EPOXY-NOVOLAK RESIN MATRIX (RT)

Property	Value
Density	0.044 lb/in. <sup>3</sup>
۵	27 × 10 <sup>-6</sup> in./in./°F
$\mathbf{F}^{tu}$	4.184 ksi
$F^{tpl}$	2.92 ksi
$\mathbf{E}^{t}$	$0.487 \times 10^{6} \text{ psi}$
vt	0.31 (at 5000 µ-in./in. strain)
Fcu	23.52 ksi
Fcpl	8.88 ksi
Ec	0.560 $ imes$ 10 <sup>6</sup> psi
γC	0.387 (at 5000 μ-in./in. strain)
F <sup>su</sup>	].54 ksi
G	0.191 ksı

### APPENDIX C

### SPECIFICATIONS

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## GENERAL SPECIFICATION | LAMINATE ORIENTATION CODE

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Date: March 19, 1970 Prepared by: G. Wolfe Approved by: Arimee

## SwRI-S3-101 GENERAL SPECIFICATION

#### Laminate Orientation Code

## 1.0 Purpose

The purpose of this specification is to establish a Standard Laminate Code that will provide the user with a clear, concise, and common notation when dealing with Laminated Composite Materials.

## 2.0 Applicable Documents

"Structural Design Guide for Advanced Composite Applications," First Edition. Section 1.5.

## 3.0 Scope

This specification presents only the sections of the Standard Laminate Code that are applicable to the work being done presently at the Institute. For the complete code and a condensed code, see the document referenced above.

Note: This specification is intended to be used in specifying laminate orientation. It does not imply any preferred laminate design.

## 4.0 Standard Laminate Code

The Standard Laminate Code is used to describe a specific laminate uniquely. It is most simply defined by the following detailed description of its features.

#### 4.1 Standard Code Elements

- Each lamina is denoted by a number representing a. its orientation in degrees between its filament direction and the X-axis (principal axis).
- Individual adjacent laminae are separated in the code b. by a slash, if their angles are different.
- The laminae are listed in sequence from one laminate с. face to the other, with brackets indicating the beginning and end of the code.
- d. Adjacent laminae of the same angle are denoted by a numerical subscript.
- A subscript T to the bracket indicates that the total e. laminate is shown.

Laminate 45 0 90

Code

90 30

 $[45/0/90_2/30]_{T}$ 

#### Positive and Negative Angles 4.2

When adjacent laminae are of the same angle but opposite in sign, the appropriate use of + and - signs is employed. Each + or - sign represents one lamina and supersedes the use of the numerical subscript, which is used only when the directions are identical. Positive angles are assumed clockwise:

Laminate	Code
$ \begin{array}{c c}     45 \\     0 \\     -60 \\     -30 \\   \end{array} $	[45/0/-60 <sub>2</sub> /30] <sub>T</sub>
$ \begin{array}{c c}                                    $	$\left[\frac{+45}{+}30/0\right]_{\mathrm{T}}$
$ \begin{array}{c}     45 \\     45 \\     -45 \\     0 \\   \end{array} $	$\left[45_{2}/-45_{2}/0\right]_{T}$
$ \begin{array}{c c}     45 \\     -45 \\     45 \\     -45 \\     0 \\   \end{array} $	$\left[ (\pm 45)_2 / 0 \right]_{\rm T}$ , or $\left[ \pm 45 / \pm 45 / 0 \right]_{\rm T}$
$ \begin{array}{r}     45 \\     -45 \\     -45 \\     45 \\     0 \end{array} $	$\left[\frac{\pm}{2}\frac{7}{4}\frac{5}{0}\right]$ T
$ \begin{array}{r}     45 \\     -45 \\     -45 \\     45 \\     +5 \\   \end{array} $	(++++45) <sub>r</sub> or
45 -45 -45 45	$\left[\frac{+}{+}\frac{+}{+}\frac{+}{+}\frac{+}{+}\frac{+}{+}\frac{+}{+}5\right] T$

Note that, in condensing signs, the sign of the center lamina

of an odd number is left uncombined.

## 4.3 Symmetric Laminates

Symmetric laminates with an even number of laminae still list the laminae in sequence, starting at one face, but stopping at the plane of symmetry instead of continuing to the other face. A bracket subscript S indicates only one-half of the laminate is shown:



Symmetric laminates with an odd number of laminae are coded the same as even symmetric laminates, except that the center lamina, listed last, is overlined to indicate that half of it lies on either side of the plane of symmetry:

Laminate

Code



## 4.4 Sets

Repeating sequences of laminae are called sets and are enclosed in parentheses. A set is coded in accordance with the same rules which apply to a single lamina:



on the other hand:



Laminates are often composed of a single repeated set. When it is desired to refer to the laminate in a generic sense, or when the number of sets has yet to be determined, as in the sizing stages of design, the coefficient n will be used with the bracket subscripts T and S instead of a numerical coefficient.

## 4.5 Quasi-Symmetric Laminates

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Laminates which would be symmetrical about the center plane, except that the halves of corresponding pairs of laminae are of different sign, are said to exhibit quasi-symmetry. These are coded in the same manner as symmetrical laminates except for the introduction of the bracket subscript Q in place of the subscript S. The direction of the positive angle is assumed clockwise:



## SwRI 03-301

## PROCESS STANDARD FOR BORON/RESIN COMPOSITE LAMINATE FABRICATION

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#### SwRI 03-301

## PROCESS STANDARD FOR BORON/RESIN COMPOSITE LAMINATE FABRICATION

<u>SUBJECT</u>: Manufacture and Quality Control of Advanced Composite Laminate -Fiber Glass/Epoxy and Boron/Epoxy.

<u>SCOPE:</u> This process standard establishes the procedures for the fabrication and quality control of laminates of fiber glass/epoxy and boron/epoxy composites to be used in the evaluation of simple bonded joints.

#### REFERENCES:

- Division X "Processes and Effects," <u>Structural Design Guide</u> for Advanced Composite Applications, Final Draft, November 1968.
- (2) "Structural Airframe Application of Advanced Composite Materials," Volume VII, <u>Manufacturing Methods</u>, by B. E. Chitwood and J. R. Stovall, The Fort Worth Division of General Dynamics, Technical Report AFML-TR-69-101, May 1969.
- (3) "Advanced Composite Wing Structure," Grumman Aircraft Engineering Corporation, AF Contract F33615-68-C-1301, First Quarterly Progress Report, May 1968.

### MATERIALS AND EQUIPMENT:

- (1) NARMCO 1581-5505 preimpregnated glass fabric.
- (2) NARMCO Boron/5505 epoxy preimpregnated material.
- (3) 181 and 120 dry glass fabric.
- (4) TX-1040 glass fabric (Teflon<sup>®</sup> treated) Pallflex Corporation.

- (5) 0.001-inch Mylar<sup>®</sup> film duPont.
- (b) Coroprene (rubber-asbestos) Armstrong Cork.
- (7) Herblease (EXL-1894/10% Vydax AR) Mitchell Rand/duPont.
- (8) M and N 50-ton Hydraulic Press (350°F).
- (9) Air-circulating oven (500°F).
- (10) Molds.

#### PROCEDURE:

- A. <u>General</u> Requirements for fabrication of glass fabric/epoxy and boron/epoxy composites are as follows:
  - Temperature and Humidity Control the layup area shall be maintained at 70°F ± 5°F and humidity shall be 65 percent or less.
  - (2) The number and orientation of plies in each panel shall be as specified in process instructions. The ply orientation shall be accurate to ±0.50°. Butt joints of 3-in, boron tape shall be staggered 0.500 in, ± 0.030 in, between plies. During layup, the fiber spacing of the tape shall be inspected. Loose fibers, crossovers, and gaps greater than 0.030 in wide shall be repaired. All discrepancies shall be noted and approval for further use must be obtained from the project leader.
  - (3) To avoid penetration of boron filaments into the flesh or clothing, the following safety precautions must be used:
    - (a) A coat of nylon or equivalent tight-weave, smooth surface fabric must be worn at all times when working with boron/ epoxy materials.

- (b) Safety glasses or eye shield shall be worn when cutting boron filaments.
- (c) Immediately remove any filament which penetrates the flesh to prevent the filament from breaking off or penetrating deeper.

## B. Laminating Requirements

- (1) Preparation The steel tooling plate shall have two coats of EXL/10 percent Vydax, each coat allowed to dry at least 10 min, followed by buffing. If steel restraining dams are to be used for boundary supports, they should also be coated with two applications of EXL/10 percent Vydax, Steel dams shall be 1/2 in. in width and within +0 005/-0.000 in, of the final laminate thickness. If Coroprene dams are used, 1/8-in, thick Coroprene shall be used for 8 through 19 plies of boron/epoxy or half that number of glass fiber/epoxy plies.
- (2) Laminates to be subsequently bonded shall have a peel ply which is to be placed on the tool surface. If a partial peel ply is to be used, the remainder of the tool surface shall be covered with Teflon<sup>®</sup> film of the same thickness. Weight of the peel ply shall be recorded.
- (3) The lot number and roll number of the boron/epoxy or fiber glass epoxy used for the layup shall be recorded. Panel weight before and after curing shall be entered on the quality control sheet for each panel. The preimpregnated materials shall be maintained in

0°F storage until ready for use. The material is then removed from storage and allowed to warm to room temperature while sealed in the polyethylene bag.

- (4) Hand layup shall use Mylar<sup>(b)</sup> templates for each ply of material. Each template is scribed with the panel size, shape, and fiber orientation. It will be identified with the panel number, dash number, and ply number counting from the tool. Tooling pin holes will locate the template on the tool. Boron or fiber-glass material is laid up on each template, trimmed to the trim lines, and covered with transparent polyethylene film. The material is taped in place and inspected with template layup. If the panel is not laid up immediately, the material is returned to 0°F storage until ready for layup. Templates are allowed to come to room temperature before protective sheet is removed. The Mylar<sup>®</sup> template is placed (material down) on the tool next to the tooling pins. The template is rubbed over the layup to create intimate contact with the tool or preimpregnated peel ply, The template is removed by rolling from one corner with the axis of the roll perpendicular to the fiber direction. Each subsequent ply is located in the same way. The exposed material is inspected after each template is removed.
- (5) When the last ply of material has been laid up on the tool, a boundary support is located around the periphery of the panel.This boundary must be within 0.06 in. (maximum) of the edge of

the panel. It must be slightly thicker than the cured panel. Resin (liquid and at 85 psi) must not be able to leak under the boundary support or through the corners. All resin loss must be vertical. The layup is then covered with one ply of TX-1040 trimmed to fit within the boundary support. The required number of bleeder plies are trimmed to fit within the boundary support and are placed on the TX-1040, (One-ply, 120-dry glass fabric bleeder should be used for each 4 to 5 plies boron; one ply of 181 preimpregnated glass fabric is equal to the thickness of two plies of boron.) The entire layup is covered with thin  $Mylar^{(0)}$  (0,0075 in, thick) which extends over the boundary support and is taped to the tool on two opposite sides. The Mylar<sup>®</sup> film is perforated on 2-in, centers which allow escape of gas, but prevent excessive loss of resin. The layup is then covered with one ply of 181-dry glass fabric large enough to completely cover the layup and overhang the tool by at least 4 in. on at least one side. This ply permits gas in the panel to vent to the atmosphere. A press plate (or upper tool plate) is placed over the layup and the panel is cured in the hot platen press. See Figure 30 on the following page.

(6) The panel is cured at a pressure of 85 psi with temperature cycle as follows.

(a) 200°F for 2 hr

- 1. Separator Cloth (TX1040) or Peel Ply
- 2. Boron or Glass Fabric Laminate
- 3. Separator Cloth (TX1040)
- 4. Bleeder Plies (120 Glass Fabric -- 1 ply per 4 plies Boron or 2 plies Glass Fabric
- 5. Mylar Film (Overlaps Boundary Support)
- 6. 181 Vent Ply (Overlaps Tool)
- 7. Boundary Support



FIGURE 30. LAYOUT PROCEDURE

- (b) 300°F for 2 hr
- (c)  $350^{\circ}$  F for 2 hr

Allow panel to cool in press under 85-psi pressure until cooled to 125°F. If postcure is required, this may be accomplished in the air circulating oven.

(7) Trim laminate to remove resin flash and record panel weight.Calculate and record the retained resin content on the quality control sheet.

Quality Assurance Provisions

 <u>Receiving</u> - All materials shall be inspected upon receipt and before use in layup in accordance with provisions outlined in The Structural Design Guide for Advanced Composite Applications.

- (2) Process Control A quality control sheet shall be maintained for each panel to verify compliance to the requirements of this specification.
- (3) Process Verification Panels A 15-ply, 3 × 9 in., 0° orientation process verification tab shall be molded simultaneously with each panel using the same lot of material unless excess (cutoff) panel material is available. The process verification tab shall be submitted for testing for flexural strength and horizontal shear strength at room temperature prior to machining of the test specimen laminate.
- (4) <u>Nondestructive Test Requirements</u> Each laminate, including test tabs, shall be inspected ultrasonically for voids, delaminations, and missing plies and results recorded on the quality control sheet. Each machined detail shall be visually inspected for filament orientation (or other nondestructive inspection) and results recorded on the quality control sheet.

SwRI 03-304

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PROCESS STANDARD FOR ADHESIVE BONDING OF ADVANCED COMPOSITE LAMINATES AND METAL ATTACHMENTS

#### SwRI 03-304

## PROCESS STANDARD FOR ADHESIVE BONDING OF ADVANCED COMPOSITE LAMINATES AND METAL ATTACHMENTS

<u>SUBJECT</u>: Bonding Techniques for Preparation of Physical Test Specimens and Bonded Joints from Fiber Glass/Epoxy or Boron/Epoxy Composite Laminates and Metal Attachments.

<u>SCOPE</u>: This process standard establishes the procedures for surface preparation of composite laminates and metal attachments and adhesive bonding and cure of joints or other attachments.

## REFERENCES:

- ASTM-2093-02T Preparation of Surfaces of Plastic for Adhesive Bonding.
- (2) ASTM-2651-67T Preparation of Metal Surfaces for Adhesive Bonding
- (3) "Investigation of Structural Design Concepts for Fibrous Aircraft Structures," by G. C. Grimes, B. J. Pape, and
  J. H. Ferguson, Southwest Research Institute, Technical Report AFFDL-TR-67-29-Vol. III, November 1967.

## MATERIALS AND EQUIPMENT:

- (1) 3M Company AF-126-2-0, 06 Film Adhesive
- (2) NARMCO Materials Division, Whittaker Corporation Metlbond - 329 Film Adhesive
- (3) 7075-T6 Clad Aluminum Alloy
- (4) 6A1-4V Titanium Alloy

- (5) 1581 Glass Fabric/5505 Epoxy Laminate
- (6) Boron/5505 Epoxy Laminate
- (7) Constant Temperature Bath (150 to 650° F)
- (8) Air-Circulating Oven (RT-500°F)
- (9) M&N 50-Ton Hydraulic Press/Heated Platens (RT-600°F).

#### CLEANING PROCEDURES:

- A. <u>Aluminum Alloy</u> Aluminum alloys are to be cleaned prior to applying the adhesive bonding agent by the following procedure.
  - (1) Wipe with solvent (MEK)
  - (2) Immerse in Oakite 61B (6 to 8 oz/gal) at 160 to  $180^{\circ}$  F for 5 min
  - (3) Rinse with R. T. running tap water for 1 min
  - (4) Immerse in Oakite 34M (8 to 16 oz/gal) at R. T. for 10 min
  - (5) Rinse with R. T. running tap water for 2 min
  - (6) Dry in air-circulating oven at 200° F for 5 to 10 min.
- B. <u>Titanium Alloy</u> Titanium alloys are to be cleaned prior to applying the adhesive bonding agent by the following procedure:
  - (1) Grit blast with fine grit
  - (2) Immerse in Oakite 31 (1 part Oakite to 2 parts water) at R. T. for 5 min
  - (3) Rinse with R. T. running tap water for 3 min
  - (4) Immerse in the following solution at R. T. for 2 min
    - (a) 841 ml HCL acid (Reagent Grade, 37 to 38 percent)
    - (b) 89 ml Orthophosphoric acid (Reagent Grade, 85 to
       87 percent)

- (c) 03 ml HF acid (Reagent Grade, 60 percent)
- (5) Rinse with R. T. running tap water for 3 min
- (o) Air dry for 1 hr at less than 60 percent R. H. and above 65°F
   or oven dry for 15 min at 180° to 200°F.
- C. <u>1581 Glass Fabric or Boron/5505 Epoxy Laminates</u> Glass fabric or boron laminates in epoxy matrix are cleaned for application of adhesive bonding agents as follows:
  - (1) Wipe with solvent (MEK or acetone)
  - (2) Sand with emery paper or sandpaper, fine grit, no larger than No. 400
  - (3) Wipe with clean, dry cloth
  - (4) Wipe with solvent (MEK or acetone).

#### BONDING PROCEDURES

- A. <u>AF-126-2-0.06 Film Adhesive</u> The AF-126 adhesive is a nitrileepoxy, unsupported B-stage film adhesive manufactured by the 3M Company. It is used with EC-2320 Primer according to the following procedure
  - (1) Clean parts to be bonded (see Cleaning Procedures above)
  - (2) Apply EC-2320 Primer to bonding area by spray, brush or dip method
  - (3) Dry primer in air-circulating oven at 150°F for 30 min
  - (4) Cut film to be used from roll with separating liner in place
  - (5) Place film on primed part using the separating liner as a protective cover

- (b) Roll film onto part with a rubber roller insuring that no air is trapped between surface and film
- (7) Remove protective cover
- (8) Assemble parts
- (9) Cure bond at 50 ps1 and 275°F for 1 hr. Heat-up rate should not exceed 10°F/min and cool down under pressure to 200°F or below. Temperature should be monitored at glue line.
- B. Metlbond 329 Film Adhesive The MB-329 adhesive is modified epoxy, nylon cloth supported, B-stage film adhesive manufactured by NARMCO Materials Division, Whittaker Corporation. The bonding procedure is as follows
  - (1) Clean parts to be bonded (see Cleaning Procedures above)
  - (2) Apply primer to bonding area
  - (3) Dry primer
  - (4) Cut film adhesive to be used from roll with protective linersin place
  - (5) Remove paper separator and place film on part using plastic liner as a protective cover
  - (6) Roll film onto part with a rubber roller to insure that no air is trapped between surface and film
  - (7) Remove plastic protective cover
  - (8) Assemble parts

 (9) Cure bond at 50 psi and 350°F for 1 hr. Heat-up rate should not exceed 10°F/min and cool down under pressure to 200°F or below. Temperature should be monitored at glue line.

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SwRI S3-401

# TEST STANDARD FOR FIBROUS COMPOSITE TENSILE SPECIMENS

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#### SwRI S3-401

## TEST STANDARD FOR FIBROUS COMPOSITE TENSILE SPECIMENS

## 1.0 PURPOSE

It is the purpose of this standard to provide a standardized technique for measuring the static tensile properties of boron/epoxy and graphite/ epoxy composites subjected to a monotonically increasing load to failure.

## 2.0 APPLICABLE DOCUMENTS

"Structural Design Guide for Advanced Composite Applications, 2nd Edition, Sections 7.3.1 and 7.3.2.

## 3.0 SCOPE

This standard covers both boron/epoxy and graphite/epoxy materials up to 18 plies thick. Measurements shall include load/biaxial strain data to obtain biaxial stress-strain curves to failure under constant strain rate conditions.

#### 4.0 SPECIMEN PREPARATION AND INSTRUMENTATION

Specimens are to be laid out and cut from a suitable size panel to the dimensions shown on the drawing below. Subsequent to cutting out the specimens, tabs are bonded onto the specimens in groups of three or more (see drawing below). Strain gages are to be as described on the drawing.



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## SwRI Standard Tensile Specimen for Composites

#### Notes:

- 1.  $t_1$  boron/epoxy or graphite/epoxy specimen  $\leq 18$  plies thick unidirectional or angleply
- 2.  $t_2$  fiberglass/epoxy tabs 0.100 ± 0.01 in. thick (approximately 12 plies 1581)
- 3. Strain gages Micro-Measurements 06-250BF-350
- 4. Tolerances:  $X \pm 0.1$ )

 $XX \pm 0.04$  unless noted otherwise

Fractions  $\pm 1/16$ )

- 5. t<sub>3</sub> boron/epoxy 0.96 in. wide } sides to be smooth, splinter free and - graphite/epoxy 0.75 in. wide } flat and parallel within 0.015
- 6. Diamond cutoff wheel to be used in sizing specimens from panel
- 7. Tabs are bonded on in groups of three specimens or more at time with strip tabs leaving 3/8-in. spacing between specimens. Individual specimens are then sliced off by cutting through tab material.
- 8. Use stand Instron wedge grips with fine serrations.
- 9. Tab bonding: cure adhesive 1 hr at 275°F at 50 psi in heated platen press.

## 5.0 TESTING

In addition to the strain gages, a clamp on extensometer with a 2-in. gage length will be used on each specimen in order to control the strain rate during test and to provide back-up load-deflection curves should they be needed. Loading should be on a monotonically increasing basis at a constant strain rate of 0.00125 in./min. Load and strain shall be recorded automatically, either continuously or at known automatically spaced time intervals.

## 6.0 FAILURE ANALYSIS

All specimens shall be categorized as to failure type, such as (1) net section tension, (2) delamination, (3) diagonal shear, (4) brooming net section tension delamination, or (5) any combination thereof. Location of the failure shall be measured and recorded. \* Any type failure between tabs is acceptable. Any type failure under the tabs is unacceptable. Complete failure description, type and location shall be recorded.

### 7.0 DATA REDUCTION

Raw data shall be appropriately processed to yield stress-strain data from which biaxial stress-strain curves may be plotted. Proportional limits, knees, moduli, Poisson's ratio, and ultimate strengths shall be

\*Photographs of typical failures shall be made for record.

located, calculated, and tabulated along with related strains. Complete computerized data reduction, plotting, and the tabulation of data is acceptable.

## APPENDIX D

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## ULTRASONIC THRU-SCAN AND RADIOGRAPH INSPECTION RECORDS ON BORON/EPOXY ADHEREND PANELS

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FIGURE D.1 INSPECTION RECORD, B-12



FIGURE D.2 INSPECTION RECORD, B-13 D-3





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FIGURE D.3 INSPECTION RECORD, B-14

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FIGURE D.4 INSPECTION RECORD, B-15



FIGURE D.5 INSPECTION RECORD, B-16



FIGUPE D.6 INSPECTION RECORD, B-17



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FIGURE D.7 INSPECTION RECORD, B-18

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FIGURE D.8 INSPECTION RECORD, B-19



FIGURE D.9 INSPECTION RECORD, B-21


FIGURE D. 10 INSPECTION RECORD, B-22



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FIGURE D. 11 INSPECTION RECORD, B-23





FIGURE D. 12 INSPECTION RECORD, B-24 D-13



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FIGURE D.13 INSPECTION RECORD, B-25

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### FIGURE D. 14 INSPECTION RECORD, B-26



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PANEL B-28, 9 ply, 0 ± 45° 10/30/70

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FIGURE D. 16 INSPECTION RECORD, B-28

### APPENDIX E

### SELECTED TYPICAL ADHEREND MATERIAL TENSILE STRESS-STRAIN CURVES AND PHOTOMICROGRAPHS

ta kultada

### SPECIMEN T-142 (Tension) $\left[ \left( 0/90 \right)_4 / 0 \right]_T$

FIGURE E.1 STRESS VS STRAIN





STRAIN (THOUSANDS OF MICRO-INCHES/INCH)

E-2

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## PANEL B-14 PHOT OMICROGRAPHS FIGURE E.2

(d) 50N 90° Photo No. 15306



(c) 50X 90° Photo No. 15259





50X 0° Photo No. 15305 (a)



E-4

\*1bid



(a) 50X 90° Photo No. 15303



(b) 50X 90° Photo No. 15260

FIGURE E.4 PANEL B-16 PHOTOMICROGRAPHS E-5



E-6



STRAIN (THOUSANDS OF MICRO-INCHES/INCH)



(a) 50X 90° Photo No. 15308



(b) 50X 90° Photo No. 15296

FIGURE E.7 PANEL B-22 FHOTOMIC ROGRAPHS

### FIGURE E.8 STRESS VS STRAIN

SPECIMEN T-233 (Tension)  $\left[0/90_2/0\right]_{4T}$ 

<sup>σ</sup>u = 40,872 (41,672)\* psi <sup>ε</sup>u = 3,200 x 10<sup>-6</sup> in./in. <sup>ν</sup> = 0.0377 (0.0400)\* <sup>σ</sup>pl = 24,183 (24,983)\* psi

 $\sigma_{pl} = 24,183 (24,983)* psi$   $\epsilon_{pl} = 1,634 \times 10^{-6} in./in.$  $E = 14.800 \times 10^{6} (15.289 \times 10^{6})* psi$ 



E-9





(d) 300X 0° Photo No. 14861

PANEL B-23 PHOT OMICROGRAPHS

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FIGURE





E-10





(a) 75X 90° Photo No. 14881



(b) 75X 90° Photo No. 14984

FIGURE E. 11 PANEL B-25 PHOTOMICROGRAPHS E-12



E-13

\*Ibid

- J<sup>e</sup>



(a) 50X 90° Photo No. 14986

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(b) 50X 90° Photo No. 14987

### FIGURE E.13 PANEL B-26 PHOTOMICROGRAPHS







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APPENDIX F

### COMPLETE EXPERIMENTAL DATA ON BONDED JOINTS

TABLE F

# DETAILED TEST DATA ON BONDED JOINTS

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1 2		-	-		-	-	7	<u>~</u>				- N	5	~	<u>د</u>									~~~~	+			
Adheu Strew Fudure	187.4	1.38.4	1.880	1.1	1.4.20	1.681	4.679	4,59	151		3.54	3,51(	3,605	3,235	3 47	£7.	7	4.39	61 7 7	4,52	3.96	3.25	4 62	3.94	4.82	12.4	<b>1</b> .4	4'40
Adherend Strew af Fadure, pu	24,888	24,444	21,256	21,629tB1	23,511	517.52	25,889	14.945(B)	61.75(1	44 444	265.9	65.262(B)	126'11	16.067	12,056	12,015(8)	20.7-8	25.(MIL)	25.681	18,620(8)	33.333	17,333	118 81	33.166(B)	27,000	25.333	23 222	25.155(B)
Failure Load. Ibirin a	021.1	1.100	0_6	1,(16.3	261.1	9211	1,165	1.147	55. 1	1740	1.8.25	1 19.3	\$651	1,430	58571	0251	2,410	222	2.250	242.2	0.00	820	1165	566	1.215	0+1.1	1.045	1.133
Bonded Joint Area in2	0.250	52 0	- 52 -	5730	0.250	0.250	1244	0.250	AUX O	1050	0.5(0)	405 D	0.442	244.0	211 0	744	0.508	41.546	903.0	0 \$0-	252.0	0 252	636.0	0 252	1220	122.0	0.252	0 252
Overlap Length in	0.250	- 52 -	0.250	1 280	11.254	0.250	0.250	052.0	050 0 4 5	050 0 4 5	2 + 0.250	2+0.250	2 × 0 218	8120 - 2	2 × 0.218	2 + 0.215	052.0 + 2	2 + 0.250	2 + 0 250	2 1 0 250	0.250	0.250	0500	0.250	0.250	0.250	0.218	0.239
Adherend Eros/section Area, in <sup>2</sup>	11 (14.5	510.0	571 0	14151111	1 114	11114	11145	0.046(B)	1038	1.64 0	-200	0.027(B)	11 (18/4	0 (98d	1991) (1	0 (184(3)	0,4,0	680 0	\$80 **	0.089,81	0.030	030	0.0.0	0.01000	579.0	5700	571) ()	0.045(B)
II Adherend Thickness, in																					0.032	0.032	0112	260.0	540.0	11.1145	0.045	0.045
Adherend Width	1 (80)	1 (11)	1(11)	1001	665 I'	1 0:00	966.0	\$66.0	1 015	i i i	101	5101	1 013	1 014	7101	1 014	1014	1 013	\$10 I	rle l	0101	6(H) [	1 008	600 i	Star 1	016-1	0101	6001
Composite Adherend Thickness (Net)	0.047/0.045	570 0 2 10 0	0.047.0.045	10112 0 095	1 04 6 04	0.046 0.04	370 0 270 0	0.047-0.047	AL00410 v C	1 0 0 1 0 0 1 1 0 0 1 1	2 - 0.0016/0.027	$2 + 0.016 \pm 0.02^{+}3$	380 0,970 0 · 2	1 0.046 0.085	280 0.910 0 47	2 - 0.046 0.085	2 × 0.045/0.089	480.0 \$t0.0 × 2	2 + 0.045 0.082	2+0.045.0.085	0.030	0.030	0.630	010 0	945	0.045	11 (144	0 045
Composite Adherend Panel Number	B-21 A D	B-21A D	B-21 A (D	B-21 A D	H .4 × H	B-24 A B	B-244/H	B-24A B	R.I.24 (D.R.I.3K	B-12A D B-13K	B 12A D B-13K	B 12A D. B-13K	B-14J K. B-15K	B-141/K, B-15K	b-14J N. B-15K	B-14J-K, B-15K	B-241/K, B-17A	B-241 K. B 17A	B-243 K. B-11A	B 24J K. B-17A	B-186	B-18C	B.1M	B-1KC	B-16D	8-161)	B-16D	B-16D
( ompwite Adherend Faber (Prentation	10141	10101	16101	lolat	10.45.0 45.01	0.45 0.45 01	0 145 11 45 11 5	01-45/01-45/01S	2. fals, futer		2 × 101 ×15. [0] × 7	2 × [0] 31 [0] 61	1 10/500 01 × 2	1 (0 500/00) × 2 × 100/00/1	1 [0,8106/01]	1 [0/8006.01] 1 [0/806.01] × 2	2 × [0/+45,0 45/0]S [10/+45 0 45/0]S	2 × 10 +45 0 45 0 S	2 × {0.+45/0/-45/0 S [r0+45/0-45/0]S	2 + [0, +45 0 45/0]S [(0 +46 0, 45 00/0]S	[0]6T	[0] 67	10 41	19[0]	0/+45/0 45/01c	10/ - 45 0 45/015	[0/+45/0/ 45/0]S	[0:+45'0 45'0]S
Adherend	8-8	H-H	8-8	x x	x z	H H	ЯH	вн	H H	8	- 20	8	H-H	H-H	H.H.	e.a	8-8	6-B	8-8	B-B	B-T	B T	1-8	B-T	B-1	B-T	B-1	B-T
Adheure Thickness	0100.0	5000.0	0 0015	0100'0	0.04030	0.0035	0.0025	00.00 0	0.015	0.0024	0.0025	0.0023	0.0024	0,0020	2100.0	s.(0)/0	0 0025	\$c00.0	0.0040	0 0033	0.0015	0 0040	0.0010	0.0022	U 0050	\$100.0	0.0015	1200.0
udheave	1 126 2	1 126-2	1.126-2	124-2	1 126 2	1 126-2	1 1 2 4 2 1 F	1 126-2	1.126.2	1-126-2	1-126.2	2 421 11	1 124-2	1-126 2	1 126 2	1-126-2	11 126 2	U-126.2	4 -126 2	VF 126-2	vF-126-2	VE-126-2	VI-126-2	VI-126-2	(F-126-2	11-126-2	VE-126-2	NI-126-2
l ength Hetween Labs	4 1 / 4	4 1 4	×	1 X -	4 1 4	4 1 1 4	* *	4 1 X	7	V 117	4 4 4	7 7 7	4 4 1	411.4	ч 3.   т	4 	4 1 4		4 4	1 1 1	¥ + 1 -	2-14	2 1/4	F 1-2	2-3:6	2.3 8 A	2.3 8 ¥	2 3/8 2
Joint Type	7	7	-	;	15	15	15	1.	10	11	14	10	10	10	10	10	DL	14	10	10	S L	SL	SL	SL .	s t	S.L.	S.L	SL
Specification Number	1141			114 1 14	15421	15422	15423	154 2 442	LSA 31	23 851	E.1 4 5 1	ISA LAVE	[SA-4-1	154-4.2	1.54-4.3	ISA-1 AVE	154 6 1	1 2-4-5	15153	LS4-5 44K	L54-6-1	LSA-6-2	LSA 6-3	LSA-6 Avg	L5A-7-1	LSA-7-2	LSA-7-3	LSA-7 A.

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A: newse Stress B Failure psi	147.5	- 51 5	165 5	175.2	2.650	103	4.356	189'8	1151	4 4	4 563	1 644	× 361	1.41	101	, ō		usta	08215	1 2 2 4	4-1 1	116	7	4,1	1.325	4137	1 102	4 503	
Adherend Strew at Failure psi	91.0176B1 82.615(Ti)	45.514(B) 40.615(T))	56.972(B) 76.000010	(11)[74]6.	050.21	22.662	32.0.25	20.420(8)	- 10 57	182.42	1.1.1	26.55°(B)	1 36 7	10.526	10.276	11,024	10,44 %B)	755 7	10 534	19.719	10,474(B)	12.326	03- 34	145 11	30.550tB)	0. 5 12 1	1120 021	1 1 1 1 4	
f aulure Load I he in i	2.685	2.620	1.4.5	1921	1.350	2.055	2.215	6.81	1921	1142	2,445	3.368	00.1	1,740	1.10	02 51	542.1	650	52.6	4 ! ¢	108	2.8.15	5014 4		2.650	1.2- 3	45.5.5	05-3	
Bonded Junt Area m <sup>2</sup>	0.50%4	0.405.0	0.4426	0 4863	5605 ()	0605 0	1 5465	0605 0	0169.0	05100	3015.0	2.050	11411	*\$10 C	1151	2 0251	01407	5942 11	1382.0	51 FT 1	. 47 0	2 11211	0.001	1000	1 000	1 2700	1.0.1	1 1000	
Overlap I ength, in	1 + 1 250	052.0.2	412 0 + 2	2 + 0 239	1 + 11 250	0520.72	2 + 0.250	2 × 0.250	2 + 0.250 -	2 1 0 250	2 + 0 250	1920.12	2.013	10.0	5 150	4 10 11	150 2	1 200	- 52 e	- 57 19	- 41 1	2,005	( H1 >	20402	400	1.250	1 250	1.250	
Area, In <sup>2</sup>	4.0295(B) 0.0325(T))	110295(B) 01525410	0.025448+	0.0291(8) 0.0325(10	1 089	9060.0	244111	(Bronwin ti	2,500	11.118.435	10400	187685010	802111	1 1453	0.1665	0 1660	0.16.2.(B)	111101	1.486.1	2 saint	1816550-0	0.05	111560	2460.0	1117.02111	- 2 10 11	474010	1. 10 0	
I Adherend hickness in	910 0 - 5	1 10010	480.0 + 2	2 + 4 mbs	310015	2+0045	2 · 0 (144	540.045	2 + 11 +45	2 + 0.045	2+0.045	51111112	113	112			11.62					1.05	1 (14 3	1 (18.5	1000		-		
Adherend   Width	, 19-1	4[0]	1	410-1	1014	1 018	_ l++ l	1.0.1	1 (812	1.020	1.001	1 014	1 01	460.0	1.003	400	1 (H15 C	tion 1	7	2 (m)	1 44.2	- 57-11	400-0	. 66-11	1 446	1 016	1114	5101	
Aberend Aberend Thickness (Net)	5¢0 0	620.0	10 - 17 M	5200	11 (155	111114	1.08	55010	- 40.0	1111	(1 CINN	11111	11 166	1110	441 1		- 44	11986-11-156	4600 4600	240.02.2000	1.000 1.000	- SSU D	44111	2 MI D	11 (156)	2001100	10142014	2011/2011/2011	
Adherend Panul Number	H 15 A	N 14.4	R 15.5	H 18.4	H 15 V	H 14 1	1 1 1	R-15A	8 221	B 223	R-233	8 223	H 264 G	H 266 G	H ZNF C	H H C	H 241 C	H 214 H 264	H 233 H 264	B-255 B-265	H 23 N 261	N 177 H	R 24 M	H 250 M	H 250 M	B 201 H	B 111 B	B 216 B	
Adherend I iber Ottentation	14141	1+1	I 4 [++]	Inter	[[0 406 01]	1 10 8106 101	10 904 - 2	110/808/01	10,015F # 5F401	10 0135 11 350 015	100 445 0 4610 010	101 - 45/11 45 10 115	2 × [+0 +45 0+0 S	2 . 1(0: + + 2 . 0) C S	2 [(0 145 0)0]S	S(0(0, 540)) = 7	SiOm ces nil	500 51 01	10 402 0141	[0.445.014]	>{0n 3t+ m]	2 - {0[5]	1.1.1	2 · [0] · 2	2 • [0] • 1	16[6]	[1]1]	[1] af	
Adherend • ombination	H H	н	18	R I	H I	8 1	B. *	B·T	8.1	В-1	Ŀ.	Ŧ	B I	i.	1.	r 3	z	H H	H-H	# #	ъ ъ	H H	H I	H I	н I Н	НН	H H	H H	
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Adhesive	AF 126.2	41-126-2	41 126 2	41 126 C	41-126-2	AI-126.2	41-12-0-2	AI -126-2	AF 126-2	AI -126-2	AI -126-2	AF-126-2	WB-329	MB-329	62 - 9N	MB-2-0	A THE	MB-329	MB-324	MB-329	41.124	MB 124	MB 329	MB 329	MB-329	AI 126.2	AF 126.2	AI 126 2	
Between Labs	4	31 2.5	2.5 15	31.52	2-1-4	7	7	2 1/4	7 1 4	4	111	7	6 7 16	4 . 4			5	4 1 12	4	4 3 16	4 1 F			21.	; I ;	513		5-1/4	
Loint Lype	14	14	1 d	10	D I	10	10	7.0	14	10	10	10	1.511		1.5			W TISE	N 1.55	N 1150	N 1156	1 51	1.541	1-51	1.51	51	2 I	15	
Number	1449	2445	6443	54.5 416	1 0 45	1 7 45		314 0 41	14141	24 14 2	54 10-1	SATH AVE		54 11 1	2 1 1 4 S	5111 ALC		121.85	NA 12/2	SA-12.3	54 12 4W	54 13 1	54-132	54 13 F	54-13 4vc 1	54 :4 1	54 14 2	1-1-1-5	

TABLE F. I (Cont<sup>i</sup>d)

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	29 E E E	A 21 2 4	1-	£ \$	5	2		ň									
Adheuve Stress at Failure, p	3.23k 3.007 3.074 3.106	3.980 4.040 3.855 8.958	1255	5.14 1 S	5-34°	4,70	4,728	4,407	4.614	3,370	3,316	3,246	3.307	174.6	3.167	3.432	3,757
Adherend Sr.ewar Failure, psi	94.037 85.342 89.450 89.450	132,673 134,815 128,454 131,4454	124 H	55,595 51,761	54,445,481	\$0.26 <sup>9</sup>	\$1,555	1411	18.358¢B1	135.873(B) 131.692(T)	133.175(B) 129.0770,01	131.051(B) 127.000(T)	133.366(B) 129.258(T))	42,437(B) 46,441(T))	\$4,244(B) \$7,938(T))	93,255(B) 95,295(T))	84,979(B) 93,241(T))
Failure Load. Ibt/in.)	4.109 3.815 3.900 3.900	4.020 4.085 3.905	÷.	1999	34.7	-,160	381.7	el. 9	\$10.	4.280	561 F	511 7	· ~ [`+	4,400	4.016	4.355	4.255
Bunded Jourt Area, in <sup>2</sup>	1 2662 1 2688 1 2688 1 2688 1 2679	00101 01101 111110	л. К	1 18854	1138881	15210	sets 1	1 5225	15210	1 27(4)	1 2685	1 26-5		1 26 1	1 2662	1 2685	5-97-1
Overlap Lenkth in	1 250		2 • 11435	2+ #431	1.1.1.2.17	2 + 1 - 51	420.42	13-1-4-2	115.11 4.7	1.25	1.250	457-1	1.250	1 260	1.251	1.250	1.250
Adheren I Crow Section Area in <sup>2</sup>	0.0436 11.044* 11.044* 11.0436 11.0436	11.13.03 11.13.03 11.13.03 11.13.03 11.13.03 11.13.04 11	1.18.1	I SALO	1815, 80.0	100 K	1.484.1	104111	0.000	0.0315(B)	0.031569		0.03247(Hz	0.01456(B)	0.0456010	01125400	0.04736H1 0.044667410
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Lunipus 'e Adherend Prnel Number	8-280.0 8-280.0 8-280.0 8-280.0	8 125 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	B-144 1 B 141	H 14H = H = 51	151-H   HT: H	B 241 1 B 1 'R	H 1 H 1 H 1 H	H_1 H 4 157 H	4_1 H   1 167 H	· *	4 1 1 4	H 1.4	H-15k	k I et	4	h ini	8 161
Composite Adherend Ether Orientition	[0 -45 0 -45 0,5 [0 -45 0 -45 0,5 [0 -45 0 -45 0]5 [0 -45 0 -45 0]5 [0 -45 - 45 0]5	2 + [0] 11 - 01 61 14 10 11 - 01 61 2 + [0] 11 - 01 61 2 + (0] 11 - 01 61 2 + (0] 11 - 01 61	T to stop to 1	Lie Autorial Lie Autorial	Lin Stor of + 1	2 * 10 +45 0 45 0 15 10 +45 0 45 0 15	2 - [0 -45 0 45 0 5 [0 -45 0 45 0 0 5	2 + 10 - 45 0 45 0's	2110-0455 00-0015-0015	Piai	10,01	10.000	1.1	Sec. Sec.	(0.410.00 d) (0.5	والمتعادية المراجع	AD144 1447-00
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l ength Primeen Tabi	27 0 7 1 2 7 1 1 1 1	4316 4316 4316 4316	72 E I P	4-13-32 4-3 k	4 13 32	4.2.4	4 5 4	¥ V. 7	4.1.32	41.64	41 5 5	41.14	51 14	1.1	1-1-4	-	-
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TABLE F, I (Cont'd)

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Adheave Stress at Failure, ps	6E e't	3.958	3,783	3,893	5 14	5 : 03	5.108	2112	C80.C	142.5	- 57 5	68-	569	1	107	10/1	8971	1 462	2,488	3,036	3 108	2.8.2		1 585		105.4		141.6	182 0	2.365	2.686	2.680	2.543
Adherend Stress at Fathure, psi	131.250(B) 123.148(Ti)	131,908(B) (123,765(T))	126,316(B) 118,154(Ti)	129.825(B) 121.689(Ti)	60.554	641,146	60.793	- C1=0="1"	502.04	Sto 28	44,344,81	5-0'6	5.154	5.628(B)		21594	17,755	1811-19.01	30.969	35,439	45,015	42,130		110 44		161,925(B)		1.05 001	101 441	102.770(B)	134.323	133,493	142,105
Failure Load, Ib(/m.)	3.990	4.010	3,840	3.947	5 250	5 2040 .	5.210		520	1.050	051-	1.59	1.405	1492	210.2	(169.5	4.0-5	1015.2	9.600	11.850	12,100	11 146	•		345		1 972 •	13. 1	4 810	4.758	1,070	4,060	4,320
Bonded Joint Area, in <sup>2</sup>	1 013	1 013	ŝto t	101	1 02:0	6101	1 020		1 2005	30.5	1 3639	0110 2	2 0130	1007	0.011	2 4444	1 1 4 2 1	19969	à 5833	3.9038	4244	1 40.00		ē	4. 4.	ATTN 1		. 10 .	2 020	2.012	15150	08181	1 > 1 45
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Adherend CrowSection Area, in-	0.0304(3) 0.0324(1))	0.0324430	0.0304cB) 0.0325cTo	0.0304(8) 0.03247(1)	"440 ti	49.40 (-	0.0%5		0.0850	0.48°33	0.0663481	2541.0	5221 0	01-10 B	1 144	-1410	1 26.33	1810202 11	0 2013	0.2605	14410	0.2515 0.2525(B)				0.04 (B)	et other t	5 11 T 1 T	-11465	J 046 4(B)	0.040.	2112 11 11	10.01 2014
II Adherend Thickness, m	4100 - 2	400 47	4100.57	900	2 - 0.045	510012	5 - 1045 5 - 1045		51010	She i st	210.012	n 1sa			1 160		25.6	1 26.	1204	77	7-27	1 2	•				•				•		
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Composite Adherend Thickness (Net)	01010	0.430	11 48 323	010	1.055	S \$ \$ \$ 11	0.054			1.15	1.44.1	*		-2-14	1. 1.4	142 1		- 2 44 1	0.260	· 241	1				100,000	0.046.1 0.0460.0	. 440.0	479.0	11(11)	440.0	2+ 0.015 0.032	2 + 0.03 × 0.031	1000 0000000
Composite Adherend Panel Number	R 151	8.151	H 121	H 14	H 150	1 1 4	B 150		8 1 °h	N-14	K 1 . K	H 230 L	8-2311	8.201	* 		R 2.1	27 1	भ <b>ु</b> भ	1.5	た <i>1</i> エコ	4 X #	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		1	R.26. 1		B 240 H	H GF2 H	R 24D H	NUT N ICL N	B LOH N. B LVA	REDUKT REFER
Composite Adherend Fuber Oraentation	10161	10161	14101	19101	L to stop of	1 10 4006 111	[10 405 01] [10 607 011	101 - 45 11 - 45 6 - 010	10 - 45 0 - 45 10 615	210 0125 11 25- 10	10 - 4' 0 - 45 10 B	1510 200/01 - 2		1748.006.01 + 2	* 11 406 01 * F	1 1 0 606 01 1 1	110 40 01 11	It to Can of + 2	Si0te stead + 6	2 - [10 - 4 - 00]		Sillion of a nilling			10441	10 [11]	15 4 25 0 1	10 57 0 57 01	10 57 0 57 0	10.51 9.51.01	2 · folat fola		191.1 181.1 × 2
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Adhesive Thickness	\$600.0	0 (NUG	2000.0	0.0003	0.0020	11012	1001		0.001	20610	0200-0	o tro	510.0	1 10 0		0400.0	>100.0	010010	Symple	0.001	12000	1360 O	. 10,0110	02000	0.0030	0,004,3	0 10110	11 (1914 5	01440	10.0442	0500.0	1 (11)	
Adhesive	AI 126.2	AI 126-2	AF 126.2	4-126-2	AL-126.2	41-120-2	AL 126-2	21-126.2	AI 126.2	AL 126.2	AI -126-2	MB-329	VB-124	WB 324	AL 126.2	AI 126.2	VI 126-2	4 10 2	AL 126.2			41 126 2	41 126.2	1 1 1 1 1	AL-126-2	AL 126-2	M 126-2	41-126-2	A1-126.2	AI 126.2	41-126-2	1.001 14	
l ength Between Tats	212	с: Тс	-	2 1-2				. 11 16	11.16	2 11 16	4 11 2	4. 4	5 L 5	e	4 7/8	N., 7	* * *	· · ·	4 II 30	41115		11 11 5		4 5 -	8.5 6	1 S X	× 1. 1	P. 3	6-7/8	8 9	4 . 		
Joint Dype		ī	1 (1	10	11.	<u></u>		10	ī	10	Ξ	1.51	1151	1-Sr 1	2.511	1 1 2 2	2.541	1 8.2	1 18 2	115	1.5	115		7	15	7	5 I	21	1	15	10		
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TABLE F. I (Cont'd)

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Hailure Type :				-				10 - L 1000						-		
-			(anto <b>-</b>		<u> </u>				1.7			ł			1	
Adheave Stress at Future pa	4 0.88	3,784	4,127	000.4	915.6	3,044	722	3,443	2.472	2,469	2,469	2.470	2.6(19	2525	2,460	1622
Adherend Stress at Failure, pu	70,454	65,266	65,210	(B)776,8A	47.4.78	84 ,560	1(4,824	95,621(B)	130.312(B) 130.512(B)	134,516(B) 1.30,312(T))	134,355(B) 130,156(T))	133.061(B) 130.260(TD	113.528(B) 116.040(T))	107.521(B) 112.278(Ti)	106.926(B) 109.292(T)	1(0,325(B) (iT)7537(Ti)
Fathure Load.	6.2(6)	05.'5	52.7.45	868.5	068.8	564.	4.4.4.4	51. 8	4.170	4.170	4.165	4.168	5.245	\$10.5	1)+6.+	5.087
lour Jour Area, m-	1 5165	56151	blei t	14760	25122	2.5275	25325	2.5308	1 6570	1 6387	1 68-0	1 6876	0010 7	2 0100	2.0080	E MM Z
Overlap Length. in	2 < 0.750	2 + 6 26	- 4 0 55 -	N: L 0 × C	2+1250	2 × 1 250	051 + 7	2 × 1 250	1 6.8.7	1 6.8.7	1 65 7	1 6.87	Cultury	2 (00)	1.000	0007
Section 4. m2	0880.0	1880 0	0.0481	18120550-0	2160.0	01910	216.2	0.09134B)	0.0320(B) 0.0320(Tu	0.0310(B)	0.0320(B)	0.03133(B) 0.0320(T0	0.0462+B) 0.04520.4	0.0472(B)	0.0462(B) 0.0452(Fi)	0.0465-8) 0.0452(TD
II Adherene I hu anew.	3								2400	210.0	210.0	210.0	0.045	5 (14 5	0.045	514) (1
Adherend Width	12	CI0-I	1 013	Ĩ	etu 1	1101	Cle 1	2101	1 000	1 10	000-1	(10k) 1	\$00 t	i noș	1 004	1 cMas
( omposite Adherend Thickness (Net)	- 80.07-10.0.4	2 + 0.047.0.087	2 × 0.047 0.087	2 × 0.04 ° 0.05	2 • 0.045 0.059	2 × 0.045 0.090	2 + 0.045 0.059	2 × 0.045-0.0697	0.032	1 0 0	160.0	11800	11 (14.6	, <del>7</del> 11 cl	11 11 11	1 (146 -
Composite Adherend Panel Number	P-141, M. B-19K	301 8. W. 141-8	B 141 W B-14K	B-141 M. B-14K	B 164 (G. B-22L	B 164/G.B-221	B 164 G 7.221	B-16A B. B-22L	B-15J	B-1x	8-182	b-18J	H 24M	MP2 H	N 62 H	B-24M
Composite Adherend Fiber Orientation	2 + {(0'90)4/0] [0/4094/0]	2 * [(0/8008/0)] T	E [0] 8006/00] 1 [0] 7006/01] × 2	L[0,6'06/0)] I [0,7(06,0'] × 2	2 × 10/+45/0 45 0/015	2 × [0/+45/0/ 45/0]S [(0/+45/0/ 45h0 0]S	2 × [0 +45 0 45/0]S [(0/+45/0/ 45)Q/0]S	2 × [0:+45/0; 45/0]S {(0:+45/0: 45)Q(0]S	[n] 61	14141	. 15 0]	: 19[0]	a +45 0 45 0  5	0 - 45 11 45 11	0 + 45 0 45 0]S	10 +45 0 45 01S
Adherend Combination	н.н	В·В	H-H	B-H	8-8	H-H	8-8	8-8	1-e	1.4	1 8	1 H	l ż	- ÷	1.8	B-T
Adheuve	 ت	0 0012	\$100.0	6000 U	2 <b>004</b> 2	0 0035	0.0034	0.003	3E00.0	5 <b>51</b> 00 m	\$ <b>†</b> 101 (1	0.0042	0.0035	27 HE ()	0035	0032
Adhesive	2-921-14	AI 126-2	AI 126-2	AI 126-2	AF-126-2	AF-126-2	AF-126-2	AF-126-2	AI-126-2	4F 126-2	AF-126-2	AI 126-2	41 126.2	AI-126-2	AI-126-2	AF 126-2
Length Between Taby	4-5-P	4.5/8	4-5.8	4.5.15	5-3/16	5-3/16	5.3 Ib	5-3/16	3.5.1	3-5-18	3.5.8	1.5.1	11276	9.2.8	4 - 1	3-7/8
Joint Fype	ц С	1 d	DE	Τd	DL	D.L	10	٦d	S L	S 1	2 T	15		1 \$	15	15
Spect Vation	LSA 30-1	LSA 30-2	LSA-30-3	LSA-J.) Avg	LSA-31-1	LSA-31-2	LSA-31-3	ESA 11 AVE	1 St. 12 1	· LSA 2.2	E \$4-32.3	LSA-32 4vg	155 33 1	L54-31-2	154 33.3	LSA-33 Avg

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•Fadure 7, pe											27 20 46	20 5 40	20 20 25 21 25 25		110 110 217 217	(); ; ; ;	43 49	10			10		1 1	00 40	14 67
-							+-				2	Ξ	2 2	-		-	*	55	56		5	2		Ģ	5
Adhenve Strets at Failure, p	2.21	2.227	2.229	2.222	1001	1.927	170,4	3,686	1,893	3.431	5.8.5	512 4	5,966	110	1.1.1	3,961	412.4	1 *05	1.736	1 * 74	554 1	1 1 1 1	4 4 6 3	1-5 7	4 266
Adherend Stress at Failure, pg	110.596(B) 103.727(Ta)	111.258(B) 104.348(Ti)	111.258(B) 111.348(B)	111.037cB) 104.141/T()	~2.196 67,090 69.415	59.564(B)	114,397(B) 113,135(T))	103.571(B) 102.428(T))	109.264(B) 108,159(1)	109,111(B) 107,90,7(T)	57.5-69	57- 74	681_1834	1	170.20	52.752	26.3454B)	14 5.34	24465	21 230	20.57681		174 74	\$81.1	61510.04
Failure Load. Ib(/in )	3.340	3.360	3,360	3,353	6.190 5.675 5.935	5.930	10.250	085.5	111 4 10	0.0	12.150	12450	05-01	1 475	1 1411	0 0		445	100 ×	410	144	1 17 1	242 5	1.5.2	2,165
Bonded Joint Area, in <sup>2</sup>	1 5 1 0 5	06051	1 5075	1 5090	1 5105 1 5105 1 5090	60151	2.81-5	5-1-57	0023-2	251+3	2.441	2 11444	1717	1 1628	****	0.2550	1751	51050	01050	1 5 10 5 11	1050	1.1.1.6	1 1 1 - 1	5-0-13	2.000
Overlap Length. m	2 + 4 750	2 - 0 750	2 + 0.750	2 . 0 "50	2 0 750	05.0.42	052.1.+2	2+1250	1921-2	1 251 - 2	2062	3447	1747 -	10.201	115 11	1.25		11 5441	(4.5.00)	14 6 141	0.5.40		10.00	11-20-42	11-12-0
Adherend Crow-Section Area, m <sup>2</sup>	0.0302(B) 0.0322(T))	0.0302(B) 0.0322(T)	0.0302(B) 0.0322(Ti)	0.0302(B) 0.0322(Ti)	0.0856 0.0846 0.0855	0.08523(B)	0.0906410	0.0896681	(ID_DATE)	0.08967cB1 0.09067cT0	52.81 11		1 1 4 1 1 1 1	11 (14115	A 171 -	5070 0	(1)(1)(R)	11.1143]	1000	12 240 40	181-240-0	1 1214	1111	011124 0	141522011
TI Adherend Thickness, in	2 × 0.016	2 . 0 016	2 . 0 016	3 • 0 016	2 + 0.045 2 + 0.045 2 + 0.045	5100 42	2 H L V	2 + 0 045	546-0-4-0	541111	414	1.1.													
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TABLE F. I (Cont'd)

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Adherend Stress at Failure, pu	14,764	19.694	17.943	13,800(B)	101-107	19.821	24,200	22.508	15 878	97.52	25.742 25.625(B)	12 556	22.581	21,198	FLL 48	0.0	91,115	-0	1.638	31,912 21,912		0.7.11	110.12	" JOON BI	1:5	· .35 4	67.64		040,94	7 7 7	44,542184
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	Specification Number	Joint Type	Length Between	Adhesive	Adhesive Thickness	Adherend	( omposite Adherend	( omposite Adherend	4 omposite Adherend	Alderend	II Adherend	Adherend Crive Section	(bverlap 1 eneth in	Bonded Joint	Load.	Adherend Stress at	Adheave Stress at		- 1 stu	ic lype	
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$ \begin{bmatrix} 53,5427 \\ 54,577$	LSA : 64.1	51	4 7/16	MB-329	0.6040	R I	10 - 45 0 45 015	H 24M		1 101 2	1145	0.0436	- 1 - 1	1 4567	1900	11 964	1 346	-	1	**	:
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$ \begin{bmatrix} [54,661] & D1 & [1,18] & MB [39] & 0.004 & B1 & [10] \\ [54,602] & D1 & [1,18] & MB [39] & 0.006 & B1 & [10] & [1,16$	LS 1.59 AVE	1s	4 	WB-224	6 00 3 -	d H		N A A	- 11 C	+ 10   1 - 1	1111	0.435.69		1-571	0000	51.606 45 029(P)	1,438	र- म	1 10	05 <del>2</del> 0	v. 1
$ \begin{bmatrix} 53,500x \\ 54,500x \\ 54,50x \\ 54$	1.54-66-1	īd	4.2.4	MB 329	1001	14	19 01	k 1 4		1.41	4100 + 2	1020-0	1050-2	1 0030		41194	1.206	27	. 65	2	~
$ \begin{bmatrix} 153 - 60 \ \text{vr} & D1 \\ 1.3 \ \text{MB} 32 \\ 1.3 \ 1.3 \ \text{MB} 32 \\ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \\ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \\ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \ 1.3 \\ 1.3 \ 1.3$	L5A-60-2	ī	4 9 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	MB-329	4000	 	14[11]	7 J		1 001		10200		02.001	041.	4.1.408	977	5	8	15	s.
$ \begin{bmatrix} 53 + 61 \\ 1 \end{bmatrix} \begin{bmatrix} 1 \\ 1$	LS4 60 415	ĩc	4 1.1	MB-324	1 200 4	1 8	19		le e	1.001	4 1 - 1	111111111111	0450.42	10.001	1 3 10	41.18(B)	1.126	# Fl	2 5	11	vi 7
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	LSA-61-1	D1	4	MB-329	0.0060	1.1	· I tu Muo mi	171	* 10141 *	li i		1 140.	2.00.5	. 4. 2	1 41	110 75	6651	ľ		ř	
Control D1         Columba         Distribution         Distribution <thdistribution< th=""> <thdistribution< th=""></thdistribution<></thdistribution<>	15461	<b>1</b> d	915	MB24	11.1072	1 4	L III MUD (III)	H 141	1 194	1.415	540000	10100		1 35.14	5 1 5 4	205115	0121	1	4	រន	
ISAA:1     D.1     1     MB339     noted     H     Int - 450 dig     H 238     noted     1 mm     2 mm<	15441 W	10	4 4 9	MB-329	0.0067	- H	1 10/5005 001	(7) H		200.1	210.01	Coper -	1000 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	010-1	4 641	Irr 15	1.057	12. 1		22	55
L54-21° D.L <sup>3</sup> MB-329 notes BT [01-450-450.05] B-23B nose 1 new 2+ nats nove 2+ nat 202 54.05 2.550 [0 5 90 154-52 D.L i MB-329 notes BT [01-450-450.05] B.2.R nose 1 new 2+ nats nose 2+ num 2+ 0.2 54.06 34 26.4 [0 5 92 154-52 D.L i MB-329 notes BT [01-450-450.05] B.2.B nose 1 new 2+ nats nose 2+ 1 no 2+ 0.2 54.05 34 26.4 [0 5 92 154-52 D.L i MB-329 notes BT [01-450-450.05] B.2.B nose 1 new 2+ nose 2+ 1 no 2+ 0.1 54.04 34.5 25.5 5 92 154-54.5 D.L i MB-329 notes BT [01-450-450.05] B.2.B nose 1 new 2+ nose 2+ 1 no 2+ 0.1 54.04 34.5 25.5 5 6.5 5							- -		-				-	*	-	19175	416 .			::	
1545.2 PL 1 MB32 00100 KL [0:450 45001] K.K.K. 0.09 100 Fords 1500 59.2 540 59.2 5.64 10 5 42 42 45 42 45 45 45 45 45 45 45 45 45 45 45 45 45	LS4+21*	14	~	MB-329	11-10(154	I I	5 10 0157 0 57 m	HCC H	11110	4 10 1	· + 0.044	10,000	1141	- 112	6.0.5	\$4.04%	2.520	10	× (		~
LANAS DI MERZO DUNU DI DIATATATATATATATA DI DIA ATATATATATA ANDRA DI ATATATATA ANDRA DI ATATATATA ANDRA DI ATATATATATA ANDRA DI ATATATATATATATATATATATATATATATATATATAT		2 3	•	MH 324	0.010.0	 £ 7	10.45.0.490.01×	ж	1000			100	0101	11-2	1441 5	\$12.05	1.614		v -	ş	
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### TABLE F.I (Cont'd)

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adure 3	0304	****	0 01	=		-	5	- 1		õ		1000	- 4 - 4
•												41 30 41 30	5 5 5
	• 1	2 8 2 8	9 × I	5	7	<b>6</b> 5	- 28	80	90	æ	58	~~~~	8 \$ 0 8 8 5 0 8
Adheave Stress at Failure, psi	2.271 2.290 1.920 2.160	1.095 1.196 1.336 1.209	1 755 2.048 2.141 1.981	\$1. I	1741	26E. I	1.617	1.353	1,154	1.500	1,436	1.364 1.262 1.093 1.240	748 926 973 882
Adherend Stress at Failure, pu	127,792 125,725 107,940 120,486(B)	58.228 63.579 71.125 64,311(B)	109.722 128.086 133.641 123.816(B)	5 143	59.136	921.31	54,869(B)	54,452	45.384	386.1	57,407(B)	95 781 98.750 76.575 87-135(B)	39,787 48,236 50,725 46,249(B)
Failure Load, Ibi/in i	5.150 5.205 4.350 4.902	2,760 3,020 3,350 3,043	3,555 4,150 4,150 4,012	5.26	522 3	SPETE	<b>*</b> (+6. <b>*</b>	4.770	080.4	0.5.9	5.0.3	3.065 2.840 2.460 2.388	1.870 2.555 2.450 2.450 2.450
Bonded Joint Area in <sup>2</sup>	2 2680 2 2725 2 2658	2.5200 2.5250 2.5075 2.5175	2 0260 2 0260 2 0250 2 0224	3 0330	3.0300	0740 5	05103	3 5245	3 2 350	3.5785	1551	2 2493 2 2500 2 2590	2.5000 2.5175 2.5175 2.5092
Overlap Length, in	สัยลัย	2 500 2 500 2 500 2 500	2 × 1 000 2 × 1 000 2 × 1 000	2 × 1 5(4)	2 × 1 500	005142	2 × 1 500	2 + 1 - 50	115-1-52	05-1-42	05.1.2	0000 0000 0000 0000	2000 2000 2000 2000
Adherend CrossSection Area, in2	n 0403 0 (H14 0 0403 0 0403	0 (H 74 0 (H 75 0 (H 75 0 (H 74 0 (H 74 0 (H 74 0 (H 74)	0.0324 0.0324 0.0324 0.0324 0.0324(B)	0160.0	0.0589	2440.0	0.0544.91	4.40.0	6680.0	0.0550	0.0855480	0.032 0.032 0.032 0.032 0.032(B)	1 114 ° () () (148 2 () (148 2 () (148 3 () (14 78 8)
TI Adherend Thickness, in												0.032 0.032 0.032 0.032 0.032	((600 0 ()60 0 ()60 0
Adherend Width	1 008 1 019 1 010 1 008	1 010 1 010 1 003 1 003	1012	E I	0101	rlo I	210.1	1.00	0104	1101	809 T	0.000 1.000 1.000 1.000	1 (144) 1 (144) 1 (144) 1 (144)
Composite Adherend Thickness (Net)	(UP0 0, IP0 0 (IP0 0, IP0 0 IP0 0/IP0 0 (IP0 0/IP0 0	0.047-0.045 0.047-0.045 0.047-0.045 0.047-0.045	2 + 0.016/0.030 2 + 0.016/0.030 2 + 0.016/0.030 2 + 0.016/0.030	2 + 3 (4 - 0 100	2 * 0 (.1" 0 055	2 × 0.04 ° 0.08 °	1480 0 - 10 0 - 2	2 + 0.45 0.65	2 + 0.04 - 0.084	2 * 0.04 0.082	2 + 0.04 "10.085	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	- 19 19 19 19 19 19 19 19 19 19 19 19 19 1
Composite Adherend Panel Number	8-256/L 8-256/L 8-256/L 8-256/L	8-16C/1 8-16C/1 8-16C/1 8-16C/1 8-16C/1	8-12) N. 8-131 6-12) N. 8-131 8-12) N. 8-131 8-12) N. 8-131 8-12) N. 8-131	B-14D H, B-15D	B 14D H, B-15D	B-14D H, B 15D	B 14D H, B 15D	B 16B H B 1 70	BIMBH, BITC	B 16B H. B 1 <sup>-C</sup>	B 16B H.B.ITC	4 181-4 181-8 181-8 181-8	2 2 3 2 2 2 4 8 8 2 8 8 8
Compoure Adherend Fiber Orientation	[0]81 [0]87 [0]81	10, \$\$/0 45/0 15 10 +45'0, 45/0 5 10 +45/0, 45/0 5 10+45/0 45/0 5	2 × [0] 37 ([0] 67 2 × [0] 37 ([0] 67 2 + [0] 37 ([0] 67 2 + [0] 37 ([0] 67 2 + [0] 17 ([0] 67	1[0.8(06,0)] 1[0.8(06,0)] < 2	2 × [(0.60)8/0] ]	2 [(0.90)4.0] ] 1 0.9048.0] ]	7 × [(0.408/0]]	2 × [0 +45/0 45 0]S {(0/+45/0 45 00 0]S	2 x [0/+45/0/ 45/0]5 [(0'+45.0' 45)0'0]5	2 × 10 +45/0/ 45/0]S	2 × [0/+45/0-45/0]S	1000	101-45/0 45/0 5 101-45/0 45/0 5 101-45/0 45/0 5 101-45/0 45 0 5 101-45/0 45/0 45 0 5 101-45/0 45/0 45 0 5 101-45/0 45/0
Adherend Comhination	8 8 8 8 8 8 8 8 8 8 8 8	स. क. क. क.		R R	88	B-H	H. H	9-93 19	B-B	8-4	B-B	1-8 1-8	8 8 8 4 1 4 1 4 1 4
Adhesive Thickness	0.0670 0.0070 0.0050 0.0063	0.0090 0.0080 0.0090 0.0087	0.0090 0.0082 0.0075 0.0082	0.0062	0.0072	0.00.2	0 (1069	0.0068	0 0058	0.0070	0.0065	0.0045 0.0050 0.0050 0.0050	0.0095 0.0110 0.0115 0.0110 0.01010
Adheure	MB-329 MB-329 MB-329 MB-329	MB-329 MB-329 MB-329 MB-329 MB-329	MB-329 MP-329 MB-329 MB-329 MB-329	MB-329	MB-329	MB-329	MB-329	MB-329	VIB-329	MB-329	MB-329	MB-329 MB-329 MB-329 MB-329	MB-329 MB-329 MB-329 VB-329
Length Between Tabs	****	6-3 8 6-3 8 6-3 8 6-3 8	4-718 4-718 4-718 8-7-8 8-7-8	\$ 5'16	5.5.16	\$-5 16	\$-5/16	\$-1/2	2-1/2	5-1/2	e 15	7/1-7 7/1-7 7/1-7	4.9/16 4.9/16 4.9/16 4.9/16
Joint Type	5 L 5 L 5 L	S L S L S L	0.L D.L D.L	DL	10	DL	٦ſ	DL	Π	ΡĽ	٦u	5 L 5 L 5 S 1 S 2 L	2 I 2 I 2 I 2 I 2 I
Specification Number	LSA-00-1 LSA-06-2 LSA-06-3 L5A-00 Avg	LSA-671 LSA-672 LSA-67-3 LSA-67-3 LSA-67 Avg	LSA 68-1 LSA -8-2 LSA -68 1 LSA -68 Avg	L54-69-1	LSA-69-2	£-69-5	LSA-69 AVE	LSA-70-1	LSA-70 2	LSA-70-3	LSA-70 Ave	LSA-71-1 LSA-71-2 LSA-71-3 LSA-71 Ave	LSA-721 LSA-722 LSA-723 LSA-72 Ave

TABLE F, I (Cont<sup>i</sup>d)

LSA-73-1       Dt       234       WB-329       0.0045       B-1       [10]67       B-18B       0.011       1.005       2 + 0.016       0.0322711       2 + 1.000       2.012         LSA-73-3       D1       2.34       WB-329       0.0045       B-1       [0]67       B-18B       0.011       1.005       2 + 0.016       0.0122411       2 + 1.000       2 014       4.2         LSA-13-15       D1       2.34       WB-329       0.0055       B-1       [0]67]       B-18B       0.0101       1.005       2 + 0.016       0.012741       2 + 1.000       2 012       4.2         LSA-13-15       D1       37-16       WB-329       0.0055       B-1       [0]67]       B-18B       0.0101       1.005       2 + 0.016       0.012741       2 + 1.000       2 012       4.2         LSA-14-1       D1       37-16       WB-329       0.0056       B-1       [0]67]       B+18B       0.0101       1.005       2 + 1.000       2 012       4.2         LSA-14-2       D1       37-16       WB-329       0.0056       B-1       [0]66]       B+18B       0.0101       1.007       2 + 0.016       0.0125       4.2       1.001       2 + 0.016       2 + 0.016       2 + 0.016 <th>Specification Number</th> <th>Joint Type</th> <th>Length Between Tabs</th> <th>Adhesive</th> <th>Adheuve Thickness</th> <th>Adherend Combination</th> <th>Composite Adherend Fiber Orientation</th> <th>Composite Adherend Panel Number</th> <th>Composite Adherend Thickness (Neti</th> <th>Adherend Width</th> <th>Tl Adherend Thickness, un</th> <th>Adherend Crots-Section Area, in</th> <th>Overlap Length. in</th> <th>Bonded Joint Area m<sup>2</sup></th> <th>Fadure Load, Ib/m )</th> <th>Adherend Stress at Failur : s</th> <th>Adhenve Sres - Failure pa</th> <th>-</th> <th>•Failu</th> <th></th> <th>1</th> <th></th>	Specification Number	Joint Type	Length Between Tabs	Adhesive	Adheuve Thickness	Adherend Combination	Composite Adherend Fiber Orientation	Composite Adherend Panel Number	Composite Adherend Thickness (Neti	Adherend Width	Tl Adherend Thickness, un	Adherend Crots-Section Area, in	Overlap Length. in	Bonded Joint Area m <sup>2</sup>	Fadure Load, Ib/m )	Adherend Stress at Failur : s	Adhenve Sres - Failure pa	-	•Failu		1	
L5A 73.7       D1       234       WB-329       0.005       B-1       [0]61       B-18B       0.010       1047       2 × 0.016       0.03230       2 × 1.000       2 013       4.1         L5A 73       D1       234       WB-329       0.0055       B-1       [0]61       B-18B       0.030       1 × 0.016       0.03230       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000       2 × 1.000	LSA-73-1	DL	2-3 4	MB-329	0.0058	t-sa	10) 6T	B-15B	le o o	\$00.1	2 + 0.016	0.0312(P) 0.0322(Tu)	2 + 1 000	etu 2	4.365	140.545(B) 136.180(Ti)	2,161	5 10		61	1.	- e <sup>65</sup>
[5,373]       D1       234       WB329       0.005       B1       [0]61       B-18       0.030       100       2.1000       2012       4.1         [5,373]       D1       234       WB329       0.0055       B1       [0]61       B-18       0.031       1.0       2.010       0.00236       2.1000       2.012       4.1         [5,374]       D1       2716       WB329       0.0065       B1       [0]0006       B1       [0]0006       B1       [0]0006       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.1       4.	LSA 73-2	ΡΓ	2.3/4	MB-329	0 0025	B-1	[0] 61	B 18B	0.040	1 007	2 + 0.016	0.0302(B) 0.0322(Ti)	1011.52	2 014	4.320	143.046(B) 134.161(T))	2,145			1	8	4
L5A73Arg       DL       2:3:4       WB-3:9       0.0056       B:T       [10]67       B:18       0.01310       2:0.006       2:1.100       2:0.00       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006       2:0.006<	LSA-73.3	DΓ	3.4	MB-329	5 500 U	B-1	[0] v1	B-18B	0.030	1 (016-	2 • 0 016	0.0302(8)	0001.5	2.012	515.5	148.179(B) 138.975(Ti)	2,224	10	4.	102	\$	÷05
LSA-74.1       DL       3-716       WB329       0.0055       B-1       100-901g (1)       B-154       0.085       2 + 0.455       0.0856       2 + 1.456       30210       4.2         LSA-74.2       DL       3-716       WB329       0.0066       B-1       100-901g (1)       B-154       0.088       2 + 0.455       0.0866       2 + 1.456       30210       4.2         LSA-74.7       DL       3-716       WB329       0.0065       B-1       100-901g (1)       B-154       0.088       2 + 0.455       0.0866       2 + 1.456       30210       4.2         LSA-74.7       DL       3-716       WB329       0.0065       B-1       100-91       1 + 1.05       2 + 0.445       0.0866       2 + 1.456       30210       4.2         LSA-74.7       DL       3-116       WB329       0.0065       B-1       100       2 + 0.445       0.0866       2 + 1.450       30210       4.2         LSA-74       DL       3-1116       WB329       0.0065       B-1       100       2 + 0.445       0.0866       2 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667       3 + 1.667	LSA 73 416	٦ſ	2-3/4	MB-329	0.0056	B-T	(0) <sub>6</sub> T	881-8	60303	1 046	2 × 0.016	0.0322(Tu)	0001 x 3	2 012	1.393	143,923(B) 136,439(T))	2,182	м. Ц.,	-	5	37 3	35
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	LSA-74-1	10	3.7.16	MB-329	0.0055	H-H	1 10 8106/01	N-1-8	0.084	1 001	2 1045	0 0896	0051 12	3 0210	4.205	46.931	295,1	,	~	2	2	
L3-747     D1     37/16     MB329     0.0060     B-1     [0.090,0/17]     B-15M     0.008     2.0145     0.0088     2.1150     30270     5.7       L3-716     MB-329     0.0065     B-1     [0.090,0/17]     B-15M     0.0087     1.095     2.0145     0.0868     2.1150     30250     4.7       L3-751     D1     31116     MB-329     0.0065     B-1     [0.090,0/17]     B-17     0.008     2.0145     30720     2.1150     30250     4.7       L3-753     D1     31116     MB-329     0.0005     B-1     [0.0-450,0/15]     B-17     0.0089     2.01459     2.1150     30250     4.7       L3-753     D1     31116     MB-329     0.0003     B-17     0.003     B-17     0.003     2.01459     2.11687     3.407     3.123     3.0250     4.7       L3-73     31116     MB-329     0.0005     B-17     0.043     B-17     0.043     2.0145     2.11687     3.407     3.174     3.107     3.124     3.124     3.107     3.124     3.125     3.124     3.125     3.124     3.124     3.124     3.125     3.124     3.124     3.124     3.124     3.124     3.124     3.124     3.124     3.12		1.1	0	5-9K	0 (0000	9-1	L (0/8(06/0))	B-1 < M	(10)	1 009	1 - 0145	0.0885	005 L × 2	3 0270	5.740	1.510	965 1	16,			0	f+ 5
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	L2A-14-3	10	91/2-6	671 BW	0.0060	ŀ9	1(0,90)9,01	B-154	0.088	0(4) 1	2 + 0.045	0.0858	2 + 1500	3 0270	5.045	56.813	1.667	10 - 1	•	17		
L5A-751     DL     3+116     MB-329     0.0006     BT     100-45     0.049     2+11/5     346-9     31       SA-75     DL     3+116     MB-329     0.0005     BT     100-45     0.045     2+11/5     346-9     3.1       SA-75     DL     3+116     MB-329     0.0005     BT     100-45     0.045     2+11/5     346-9     3.1       SA-75     DL     3+116     MB-329     0.0005     BT     100-1     2+0.045     0.0909     2+16-7     340-7     4.1       SA-75     AD     3+116     MB-329     0.0005     BT     100-1     2+0.045     0.0909     2+16-7     340-7     4.1       SA-55     AB     100-45     450/015     BT     0.0603     1010     2+0.045     0.1607     3+17-7     3.1       SA-55     AA     DL     311/16     MB-329     0.0065     BT     40090     2+16-7     3+27-3     3.1       SA-55     AA     AA     AA     AA     4-0015     B     1010     2+0.045     0.1090     2+16-7     3+27-3     3.1       SA-55     AA     AA     AA     4-0015     B     1     0.0603     2+16-7     3+27-3     3 <t< td=""><td>L5A-74 A.F</td><td>D.L</td><td>31/19</td><td>MB-329</td><td>10054</td><td>B T</td><td>1 [ + 8(06 0)]</td><td>B-15M</td><td>0.0885</td><td>S.4. 1</td><td>2 - 1144</td><td>0.0891495</td><td>1 1 2 • C</td><td>3 0250</td><td>1.00.1</td><td>56.128(B)</td><td>1,652</td><td>20</td><td></td><td>2</td><td>5</td><td>÷.</td></t<>	L5A-74 A.F	D.L	31/19	MB-329	10054	B T	1 [ + 8(06 0)]	B-15M	0.0885	S.4. 1	2 - 1144	0.0891495	1 1 2 • C	3 0250	1.00.1	56.128(B)	1,652	20		2	5	÷.
11.10     MB-329     0.0035     B1     [(**-45.0-45)0.65)     B.11     0.0     1.010     2.0.045     0.0035     2.1.165       15.5.75     DL     311/16     MB-329     0.0075     B1     1.010     2.0.045     0.0035     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.019     2.1.165     3.0.015     3.1.1     3.0.015     3.0.015     3.0.015     3.0.015     3.0.015     3.1.1     3.0.015     2.1.165     3.0.015     3.1.1     3.0.015     2.1.165     3.2.1     3.0.015     3.1.1     3.0.015     2.1.165     3.2.1     3.0.015     2.1.1     3.1.1     3.0.015     2.1.165     3.2.1     3.1.1     3.0.015     3.1.1     3.1.1     3.0.015     3.1.1     3.0.015     3.1.1     3.1.1     3.0.015     3.1.1     3.1.1     3.0.015     3.1.1     3.1.1     3.0.015     3.1.1     3.1.1     3.0.015     3.1.1     3.1.1     3.1.1     3.0.015     3.1.1     3.1.1 <td>LSA-75-1</td> <td>DL</td> <td>3-11-16</td> <td>MB-329</td> <td>0.0068</td> <td>B=T</td> <td>1(0. +45 0/ 45 k) 015</td> <td>l'[.8</td> <td>0.089</td> <td>1 (1)0</td> <td>24 0.045</td> <td>10184</td> <td>8121 42</td> <td>14060</td> <td>1.7</td> <td>011 110</td> <td>1 0 1</td> <td></td> <td>1.</td> <td></td> <td></td> <td></td>	LSA-75-1	DL	3-11-16	MB-329	0.0068	B=T	1(0. +45 0/ 45 k) 015	l'[.8	0.089	1 (1)0	24 0.045	10184	8121 42	14060	1.7	011 110	1 0 1		1.			
LSA-75 Arg         D.L         3111.6         MB:329         0.0062         H         10.6        0.6         10.11         2.0145         0.0063         5.1         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.1         5.0         5.0         5.0         5.1         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         5.0         7.0         5.0         5.0         7.0         5.0         6	LSA-75 2	1d	3-11-16	MB-329	0.0055	B 1	\${0 0157 0 547 0]	B-1"L	0.0.0	1 010	2400.0	0.000	1 1 1 1	3 40 -1	1 185	18, 740	1 26.7		•		·, ~	
L3A-23 vr     D.L     311/16     MB-329     0.0065     H     [00-45/0] - 45/Q <sup>2</sup> (0] S     B12     0.0807     [1.010]     2.0.045     0.0905/R1     3.2.2.4     3.2       1     adfresse to B15     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     3     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     4     <	LSA-YS 3	DL	3 11 16	MB-329	5 0.0072	6.4	110 -45/01 40 015	1.1.4	06411	1 010	2 + 0 (45	6060 0	1 1 58	07 5	3.860	42.464	1.133			54	n IS	
<ul> <li>allure Types</li> <li>allure Types</li> <li>Adhreve to BJ5</li> <li>A</li></ul>	ESA-75 AVE	D.L	3 11/16	MB-329	0.0065	14	S [0.0151 1151 113]	1-14	0.0897	oto 1	2 + 0.045	0 199054 81	-041 + :	3 4274	3.48	44.025(B)	2	1 Z.	-	: Ų.	. ~	
allure 1) Pes     Note     Stepped adhreerals for step lap (St. L.) joints mode by bonding up 2 or more laminates or Taj sh Adhreave to Bis     3       1     Adhreave to Bis									i i													
1     Adherwe to R/S     3     ••• I one trudhual splitting       2     Adherwe to FI     *     5     Some small solid s       3     Coherwe *     *     * Sternal sound solid s       4     Surface Resu     * Net section reason in travon in and inagrudinal splitting. boron 6       6     Other *     * Sterna sound solid s						· ailure Type	2	Note	Stepped adherends	for step lap	(St L ) joints me	de by bonding u	p 2 or more la	munates or	In sheets							
3 University     5 Objector       3 University     5 Net section fravues II       4 Surface Resur     7 Net section fravues II       5 Interfamentar     2 Net section fravues II       6 Other 3     6 String and Network II						1 Adhesi	re to B/5 3		" t ongstrudunal splat	ting					-							
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6 Other of Skinner and Skinner						4 Surface	Resin		+ Net section tensio	taynol brue n	udinul splitting	boron										
6 Other 9						5 Interlar	ninar		Net section tensio	n boron												
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### APPENDIX G

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### NONLINEAR DESIGN/ANALYSIS PROGRAM

# FAILURE/BEHAVIOR PREDICTION RESULTS ON SIMPLE JOINTS

APPENDIX G.1

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# FAILURE LOAD PREDICTION RESULTS ON SIMPLE JOINTS

LSA ZO SINGLE LAP COMPOSITE TU TITANIUM ADHERENDS, LSHE ADHESIVE

NUNLINEAR ORTHOTROPIC ANALYSIS, SINGLE LAP JOINT

ULTIMATE LUAD PREDICTION HASED ON Admesive - MX Stress, su = 7.176+03 Admesend I - MX Strain, sl = 5.766-03, st = 4.016-03, slt = 1.506-02 Admerend Z - MX Stress, sl = 1.386+05 N 845.7E N 000000 145.541 2.541 SECANT S 136600 0567 07950 THICKNESS = .0028 POISSONS RATID = .40 RAMBERG USGOUD CONSTANTS (SHEAR STRESS-STRAIN "URVE) G = . 80600 SECANT S = 3740 N VALUE = 6.318 -SECANT S ŝ 0 0450 • 3065 17251000 27+84000 27+84000 27484000 2750000 2750000 2750000 .0467 ហ ភ ADHEREND NUMBER ICORTHUTROPIC) ۵ THICKNESS NUMBER OF LAYERS RANBERG USGUOD CONSTANTS SL VS. EL SL VS. ET SL VS. EL SL VS. ELT SL VS. ELT SL VS. ELT OMIENTATIONS 0 45 0 ADHEREND NUMBER 2(ISOTROPIC) THICKNESS PUISSONS RATIO RAMBERG USCOOD CONSTANTS 8 VS. E TOTAL LENGTH = 3.2500 JOINI LENGTH = 1.2500 ENROR TOLENANCE = .025 MAXINUM ITERATIONS = 20 NUMBER OF STATIONS = 61 EFFECTIVE K = .010 ADHESIVE G-3

7.39196+00 7.39156+00 8.21506+00 31

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ALPHA = BETA = LAMJA = N RESET

AT ITEFATICN I EMRON IS 2.69E+09 Max1~UM STRESS(STRAIN)/ALLOWABLE = .001 IN ADHESIVE IN ADHESIVE . BIT IN ADHESIVE .935 IN ADHESIVE IN ADHESIVE IN ADHESIVE . BBS IN AD. SIVE TITERATION LERADE IS 1.11E-01 AT ITERATION LERADE IS 7.24E-03 MAXIMUM STRESS(STRAIN)/ALLOMABLE = .458 IN LOAD = 1255 AT ITERATION LERADE IS 2.51E-01 AT ITERATION Z EYROP IS 1.64E-02 AT ITERATION Z ERROP IS 1.64E-02 AT ITERATION Z ERROP IS 1.64E-02 MAXIMUM STRESS(STRAIN)/ALLUMABLE = .602 IN LOAD = 1899 AT ITERATION 2 EHRUM IS 3.87E-01 AT ITERATION 2 EHRUR IS 1.97E-01 AT ITERATION 2 ERRUR IS 1.97E-01 AT ITERATION 9 ERRUR IS 7.74E-02 AT ITERATION 5 ERRUR IS 7.74E-02 AT ITERATION 5 ERRUR IS 3.36E-02 AT ITERATION 7 ERRUR IS 3.36E-02 AT ITERATION 7 ERRUR IS 2.22E-02 211. AT ITEMATION I EAROM IS 3.44E-01AT ITEMATION 2 ERROM IS 1.35E-01AT ITEMATION 3 ERROM IS 1.35E-02AT ITEMATION 9 ERROM IS 3.55E-02AT ITEMATION 5 ERROM IS 3.56E-02AT ITEMATION 5 ERROM IS 3.56E-02AT ITEMATION 5 ERROM IS 3.54E-02AT ITEMATION I ERROM IS 3.87E-01AT ITEMATION I ERROM IS 3.87E-013.54E-01 1.48E-01 3.15E-01 1.77E-01 1.26E-01 9.54E-02 7.42E-U2 5.87E-02 2.926-02 2.316-02 VBLE = .9 2.306-01 1.276-01 1.276-01 7.1246-02 7.146-02 7.146-02 8.656-02 1.02 3.806-02 3.806-02 4.65E-02 3.68t-12 MAXIVUM STRESS(STRAIL)/ALLOWABL DAU = 4435 ITERATION FOR ULTIMATE LOAD Ечкон Ечкон EKrUK EKHUN ERHCH ERHON EHROH ENROH EXXUN ۳ o --- N m ÷ S 0 ~ 0 -1 N ഗഗരംത - 
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AT ITERATION 9 EMPOR IS 2.55E-02 AT ITERATION 1 FYROM IS 2.65E-02 MAXIMUM STRESS(STRAIN)/A.1. 4BLE = .962 IN ADHESIVE (.0AD = +718 AT ITERATION 1 ERKOR IS 1.73E-01 AT ITERATION 2 ERKOR IS 1.73E-01 AT ITERATION 2 ERKOR IS 9.476-02 AT ITERATION 2 ERKOR IS 9.466-02 AT ITERATION 2 ERKOR IS 9.466-02 AT ITERATION 2 ERKOR IS 5.486-02 AT ITERATION 2 ERKOR IS 5.486-02 AT ITERATION 2 ERKOR IS 5.486-02 AT ITERATION 2 ERKOR IS 2.466-02 AT ITERATION 2 ERKOR IS 2.466-02 AT ITERATION 2 ERKOR IS 2.546-02 AT ITERATION 2 ERKOR IS 2.05E-02 AT ITERATION 2 ERKOR IS 2.05E-02 AT ITERATION 2 ERKOR IS 2.126-02 AT ITERATION 2 ERKOR IS 2.126-02 AT ITERATION 4 ERKOR IS 2.126-02 AT ITERATION 4 ERKOR IS 2.126-02 AT ITERATION 4 ERKOR IS 2.126-02 AT ITERATION 1 ENROR IS 1.076-02 AT ITERATION 1 ENROR IS 1.076-02

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THE PREDICTED ULTIMATE LUAD IS 4677

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RESULTS FOR P = 4677

Tenter Mill

NUMBER OF ITERATIONS = 1 , MAXIMUM ERROR = .01071

	<pre>%*6775E+03 -6*5988E-0</pre>	4.4331£+03 -6.1838E-C	4.1949E+D3 -5.5552E-C	3.96396+03 -4.78075-C	3.7416E+03 -3.9340E-C	3.5292E+03 -3.0738E-0	3.3283E+03 ~2,2554E+0	3.1401E+03 -1.5197E-0	2.9669E+03 -8.4800E-C	2.8109E+03 -4.0843E-0	2.6758E+03 -5.7438E-0	2.5651E+03 1.5046E-0	2.4787c+03 2.5884E+0	2.4115E+U3 3.0046E+0	2.35666+03 2.8880E-0	2.3084c+03 2.5806E+0	2.2508E+03 2.2429E-U	2.2085E+03 1.9580E-0	2.1+55E+03 1.7569E-0	2.05546+03 1.5201E-0	1.9620E+03 1.4751E-0	1.8337c+03 1.1604E-0	1.6834E+03 4.803UE-0	1.5150E+03 ~b.4746E-0	1.3310t+03 -2.1558E-0	1.13396+03 -3.8288E-0	9.2492E+02	7.0586E+02 -6.2483E-0	4.7771E+02 -6.0504E-0	2.42196+02 -4.16186-0	1.4552E-10 -1.5916E+1	
T Y LI	.0	-4.3060E-02	-6.1887E-O2	-6.2878E-D2	-5.26196-02	-3.6462E-02	-1.9297E-02	-4.452E-D3	5.7272-03	1.06876-02	1.1327E-02	9.8413E-03	8.5644E-C3	8.4125E-03	9.6518E-03	1.1887E-02	1.4174E-02	I.****E-02	1.1479E-02	8 <b>6 5 5 5 5 - 0 4</b>	-2.05036-02	-5.5004E-02	-1.0319E-01	-1.h422£-01	-2.3613E-01	-3.1577E-01	-3.4421E-01	-4.8121E-01	-5.5b35E-01	-6.1745E-01	10-38855 ·· ·	
TVN	4.3656E-11	2°+++TE+05	4.82545+02	7.13596+02	9.3584£+02	1.1482E+03	1,3492E+03	1.5373č+U3	1.71056+03	1.8555+U3	2,0017E+03	2.1124E+03	2.1989£+03	2.2460E+U3	2,3207É+03	2.364l£+03	2.4167E+03	2.46405+03	2.5314£+03	2.6121E+03	2.7155E+03	2.8438E+03	2.94405+03	3.16256+03	3.34646+03	3.5436€+03	2.7525E+03	3.9716E+U3	60+399PL.4	4 .4353E+U3	4.6775E+03	
	1.7368E+03	1.3236E+03	9.284E+02	5 <b>.</b> 3614E+02	1.9437E+02	-9.4018E+01	-3.1844540S	-+-20101+02	-5.43576+02	-5.35806+02	-4.5285E+0Z	-3.2064E+02	-1.k711E+02	-8.4817E+01	-2.35785+01	-4.6158E+1)D	-2.6356E+01	-8.7043E+01	■1.8333E+C2	-3.0559£+02	-4.26465+02	+5.0071E+42	<b>-5.0136E+02</b>	<b>-+</b> .2427E+02	-c.749JE+02	-6.U639E+01	2.09066+02	5.23226+02	8.6884E+0 <b>2</b>	1.4310E+U3	1.5944E+03	
	5.9256E+03	5.8000E+03	5.6403E+1j3	5 <b>.</b> 4482E+03	5.2246£+n3	4,4590£+03	4.6787E+A3	E0+366*E*	3.9648E+D3	3,5094E+n3	. <b>2.</b> 9624E+U3	<b>2.</b> 3544E+03	1.4157E+03	1.43546+03	1.2143c+03	1.1304E+U3	1.1765£÷03	1.4573E+03	1.6894E+03	2.1453E+03	2.7677E+03	3.3500£+03	3.8398L+D3	4°54036403	4.5833L+03	4 8822E+03	5 • 1 + 4 9 E + 0 3	5.3749c+03	5.5739£+n3	5.74166+93	5.8771E+U3	
ĸ	0.000	.0417	EF30.	.1250	.1667	, 20H3	• <b>č</b> 5 0 0	C 1 5 2 .		.3751	. 2414.	.4583	.5000	6125.	6F35.	. b 2 5 J	. 656 <b>7</b>	EH06 -	0.024.	<b>2</b> T h 2 .	есан. •	. 3750	•1 <sup>1</sup> 7	6636.	1.0000	1.U417	1.0H33	1.1250	1.1557	1.204J	1.2500	

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NONLINEAR ORTHOTROPIC ANALYSIS, DOUBLE LAP JOINT

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AT ITERATION 1 ERROR IS 3.52E-01 AT ITERATION 2 ERROR IS 1.51E-01 AT ITERATION 3 ERROR IS 1.51E-01 AT ITERATION 4 ERROR IS 2.60E-02 AT ITERATION 4 ERROR IS 3.99E-02 AT ITERATION 5 ERROR IS 2.14E-02 M#XIMUM STRESS(STRAIN)/ALLOWAGLE = .712 IN ADHESIVE LOAD = 2830 FERTICN FOR ULTIMATE LOAD LOAD = 1 AT ITERATION 1 ERROR IS 2.32E-09 Maximum Stress(Strain)/Allomable = .001 in Admesive Load = 810 AT ITERATION LERROR IS 2.556-01 AT ITERATION 2 ERROR IS 6.036-02 AT ITERATION 3 ERROR IS 2.116-02 MAXIMUM STRESS(STRAIN)/ALLOWABLE = .600 IN ADHESIVE AT ITERATION 1 ERROW IS 1.15E-01 AT ITEWATION 2 ERROR IS 8.15E-03 Max1"U" STRESS(STRAIN)/ALLOWABLE = .456 IN ADHESIVE IN ADHESIVE 414. 4.51E+02 4.18E-02 3.88E-02 1.43E401 1.22E401 1.06E401 9.41E402 3.60E-02 3.35E-02 3.12E-02 1.29E-01 1.04E-01 9.21E-02 8.42E-02 7.83E-02 3.78f-01 2.32E-01 4.045-01 8.425-02 6.90E-02 6.30E-02 5.77E-02 5.306-02 4.89E-D2 L.75E-01 2.07E-01 7.35E-02 н MAXIMUM STRESS(STRAIN)/ALLOWABLE w **UWABL** AT ITERATION 19 EKROR IS AT ITERATION 19 EKROR IS AT ITERATION 20 EKROR IS MAXIMUM STRESS(STRAIN)/ALL LOAD = 3982 ERROR IS ERROR IS Ечкон Еккон ERROR Еннок Екгон ERROR EPHOR ERROR ERROR ERPOR ERROR ERROR Еннок EXROR ЕКНОН ERROR ERRUR ERROR EHROH ERROR ERROK ERROR ERHOR Еккок EKROP ERROR ERROR ERHOR ERROR ERROR ITERATION 11 ITERATION 12 ITERATION 12 ITERATION 12 ITERATION 14 ITERATION 14 ITERATION 15 ITERATION 15 ITERATION 15 ITERATION 18 ITERATION 18 աստ տ տ ж **с** ຳລຸ S 10 S م œ σ m ± ھ TERATION AT ITERATION ITERATION 11524110N 11524110N 11544110N 11544110N 11544110N 11544110N LOAC = 1964 AT ITERAT 5P51 = 0A0. **T**+9€ = LOAD ------Å۲ A T 4 ΑŢ A T ΑŢ ΔT AT AT A T A ΔT 4 4 4 A T Å٦ ΔŢ

THE PREDICTED ULTIMATE LUAD IS 4163

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COMPOSITE TO TITANIUM ADHERENDS, LSWE ADHESIVE LSA ZE TWO STEP LAP

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 N 252.4E ADHESIVE THICKVESS ... 0058 Polssons Ratio = .40 Ramaerg osgood constants (Shear Stress-Strain Curve) G = 80600 SECANT S SECANT S 134000 00 5+ 5+ 0 5+ 5+ 0 NUNLIVEAR URTHOTROPIC ANALYSIS, STEP LAP JOINT 58825000 200055882 136556451-.2620 48 .2640 .3062 E 16096000 2750000 UUUEFP EHROR TOLERANCE = .025 Maximum iterations = 20 Nummer uf stations = 31 per trfad Ffectivf K = .100 00 ADPERELD NUMBER 1 (ORTHOTROPIC) THICKNESS NUMMER OF LAYERS RAMHERG OSGODD CONSTANTS SL VS. EL ST VS. ET ST VS. ET SLT VS. EL ORIENTATIONS O 45 -45 Ð 2.0080 1.8720 5 1 ADHEREND NUMBER Z(ISOTHOMIC) THICKNESS POISSONS HATIO RAMHERG OSGOOD CONSTANTS S VS. E ហាហា ភាភា ព 5.9221E+00 5.8758E+00 3.9374E+00 3740 6.318 БЧ 8 5780. 00 •0873 .0880 SFCANT S = N VALUE = STEP GEOMÉTRY • STEP ] ALPHA = AETA = LAMBDA = V RESETTO

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### APPENDIX G.2

LEHAVIOR PREDICTION AT GIVEN LOADS IN SIMPLE JOINTS

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# LSA 20 SIMGLE LAP COMPOSITE TO TITANIUM ADHERENUS, LSHE ADHESIVE

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	× aa			U. -1 10065403	6 84356404 6 861356404	2.0955E+04	•
	с.	40+15×61*+	8.45656400	0.	6.9587E+04	2.1032E+04	•••
×	LAYFR	STRSL 1	STHST 1	STR9LT ]	STRSL 2	STRST 2	. STRSLT
5	г	4.68856+04	9.5554E+D2	.0	<b>5.</b> 3876E+04	+0+35256•T	• 0
	<b>ر ،</b>	37+36588°2	2°5424E+03	-+.40796+02	6.3455E+04	1.9583E+04	•0
	ב רי	4.64056+04 0.00006.00	9,5594E+02		6.4036E+04	1.9608E+04	• •
	רט +		C. 54 156 101 C. 57 156 10	4.4085E+0<	6.4115E+04 5.4195F+04	I.4532E+04 1.9653E+04	
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	~	4.6445f+04	4.5781E+02	0.	6.4356E+04	1.97065+04	
	æ	2.3977E+04	2.6456E+J3	-4.0495402	6.4+36E+04	1.9730E+04	•0
	ď	4 - 69556+174	4 <b>5</b> 821E+02	•0	6.4516E+0 <b>+</b>	1.9755E+04	•
×	LAYER	STPSL 1	STRST 1	STHSLT 1	STRSL 2	STRST Z	STRSLT
E E	-1	5.2nn5E+ŋ4	1.0595E+D3	<b>.</b> 0	6.0416E+04	1.8499E+04	••
	<u>ر</u> . ח		2,073E+03		6.0+24F+0+	1.8512E+0+	-
	с <del>э</del>	7.5437E+04	2.4(1776+1)3	U. 4.5679E+D2	20+3+350-9	1.85255+U+	
	ۍ.	5.20146+04	1.0596E+03		6.1586E+04	1.8551E+04	
	ا م	2.650nE+U+	2.40802+03	4.5680E+02	6.()528E+0+	1.8564E+04	.0
	~ α	5 - 222255 + 0 + 2 - 55555 + 55 + 55 +	1.0544F+03 2.4084F+03	0. •4 56836400	6.0571E+04	1.8577E+0+	
	) <b>5</b>	5.2032E+04	1.0000000	•	6.0756E+04	1.8603E+0"	•••
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	nu m	2.87575+04 5.648]£+64	3.13576+03 1.14916+03	-4.6857E+02 0.	5.7384E+04 5.7348F+04	1.7571E+04	•••
		2.87685+0*	3.1356E+03	4.6857E+02	5.7412E+04	1.7580E+04	
	<b>ں</b> .	5.5482F+04	1.14916+03	0.	5.7426F+04	1.7584E+04	•
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	×	LAYER	STRSL 1	STRST 1	STRSLT 1	STRSL 2	STRST 2	STRSLTZ
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<b>4</b> . 10 <b>4</b> . 10 <b>4</b> . 10 <b>6</b> . 10 <b>1</b> . 80 <b>6</b> . 10 <b>1</b> . 80 <b>6</b> . 10 <b>1</b> . 80 <b>6</b> . 10 <b>1</b> . 10 <b>6</b> . 10 <b>1</b> . 10 <b>6</b> . 10 <b>1</b> . 10 <b>6</b> . 10 <b>1</b> . 10 <b>6</b> . 10 <b>1</b>	1. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2	0 -+. 770+E+02 4. 7704E+02 0. +. 7704E+02 -+. 7704E+02 0.	$\begin{array}{c} 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 5. & 4 \\ 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×	LAYER	STRSL 1	STHST I	STRSLT 1	STASL 2	STRST 2	STRSLT 2
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×	LAYER	STREL 1	T ISHIS	STRSLT 1	STRSL 2	STRST 2	STRSLT 2
ь г г	ተ በ ጣ ታ ባ ወ ሶ ወ ፓ	а .	••••••••••••••••••••••••••••••••••••••	-++	<pre>* * * * * * * * * * * * * * * * * * *</pre>	1 1 1 1 1 1 1 1 1 1 1 1 1 1	
×	LAYEK	STR3L 1	STRST I	STRSLT 1	STRSL 2	STRST 2	STRSLT 2
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ж, ж	S T R S L S T R S L a . C 5 4 4 0 4 a . D 8 L 3 1 5 1 6 4 0 4 a . D 8 L 3 1 5 1 6 4 0 4 a . D 8 L 3 1 5 4 0 4 a . L 2 4 6 4 0 4 a . L 2 4 6 4 0 4 a . L 2 4 0 4 0	$\begin{array}{c} \mathbf{A} = \mathbf{A} + \mathbf{A} \\ $	8125 L 8175 L 1. 11546405 5. 12413546405 1. 020946405 5. 203966404 1. 020546405 1. 020946405 5. 242264640 1. 03436405 1. 03436405 1. 03436405 1. 03436405 1. 03436405 1. 03436405 1. 03436405 1. 0343645 1. 0343645 1. 034365 1. 034555 1. 0355555 1. 0355555 1. 0355555 1. 0355555 1. 0355555 1. 03555555 1. 03555555 1. 035555555 1. 0355555555 1. 035555555555 1. 03555555555555555555555555555555555555	STPSL 1 1.07466+45 5.49576+44
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13 <b>- 3,65/254</b> 0 10 0	L.44LSC4LSC4L	1.0100101 10100101 10100101		10+30541.1	9,8460F+14	u m	
12 0.	5.2869E+0	2 • 6 3 3 5 E + 0 +	•	1.1442E+04	3.9163E+04 3.9463E+04	n	4 <b>6</b> ,54
2 STRSLT	STRST	STRSL 2	STRSLT 1	ST4ST 1	STRSL 1	LAYEr	×
J2 G.	6.2999£+0	3.14046+04				17	
	1.7678E+0	1.6009E+04				16	
JJ 4.12585+U	1.75/85+0 6.29995+0	+0+3+0+0+0+0+0+0+0+0+0+0+0+0+0+0+0+0+0+				- 5 T	
0.	6.2439E+0	3.14046+04				n : 1	
13 -4.1258E+0	2. C C C C C C C C C C C C C C C C C C C	1-6009E+04				12	
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uc U. 13 =4,1258F+0	0.643467U	1.60096+04	• •	1.09496+04	3.5528E+04	4 N	
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)3	1.9604E+0 7.0137E+0	1.7831E+04 3.4482E+04				11	
12 0.	7.01376+0	3.49825+04				15	
JE U. J3 4.4056E+0	1.9504E+0 1.9504E+0	3,73316+04 1,78316+04				) <b>*</b>	
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	+ L	4 - 74 505 + 0 +	10+3+0FE • T	<b>.</b>	4,4171E+03	1.1097E+D3	2.8818E+U2
	"n.		1.32725+04	•	1.94485404	3.9080E+02	•
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	E T				4.41/1C+03	1.10975+03 2 00005+03	-2.8818E+U2
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×	しょどぶん	ST4SL 1	STRST 1	STRSLT 1	STRSL 2	STRST 2	STRSLT 2
5 t S E •	٦	+0+32<56 +	1.51646+04	•	1.0643E+04	2.14066+02	.0
	æ	4 34436404	1.5141E+D4	<b>.</b>	5.4279E+D3	6.1187E+02	-1.6722E+02
	m	+0+32769.4	1.5118E+04	••	1.0643E+04	2.1476E+D2	.0
	÷ 1	+ 0+14+14+0+	1.5095E+04	•••	5.4279E+03	6.1197E+02	1.6722E+02
	л.	4 4 4 7 7 5 F + C +	1.5072E+04	••	1.0643E+04	2.1466E+D2	•0
	р г Q	4 . 4 . 4 7 F 4 0 F	1.5043E+04		5.42795+03	6.1187E+02	-1.6722E+02
	• 0		1.50365404	<b>.</b>	1.05436404	2.14U5E+D2	
	r 0		10121111111111111111111111111111111111			6.1187E+02	1.6722E+02
	• -		16130001°T	• •	1 101401-1 	<pre>&lt;.I+U5E+U2</pre>	
	11				1.0543E+04	2.1404F+02	1.0/2254UC
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	ן <del>ל</del> - ויי				5.42746+03	6.1187±+02	1.67225+02
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	ſ.	5.63336+04	1.72445+04	• 0	• 0		
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COMPOSITE TO TITANIUM ADHERENDS, LSHE ADHESIVE LSA 26 TWO STEP LAP

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NUMLINEAR DATHOTROPIC ANALYSIS, STEP LAP JUINT

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|----------------------------------|---|-------------------------------|-----------------------------|-----------------------|---|---------------------|
| ¥                                | TAU                                     | SIGMA                         | TXN                         | TXW                   | NXS                                     | 5XH                 |
| STED JUNGE                       | <b>1</b><br>a                           |                               |                             |                       |   |                     |
|                                  | 5.4P326403                              | 2 <b>.</b> 77465+03           | 1.1642E-10                  | -4,0444E-14           | 8.3400E+U3                              | -3.6622E+D1         |
|                                  | E L + L I - Z - S - +                   | 1.**+56+03                    | 4.5213E+U2                  | -9-34546-01           | 7 <b>.</b> 9374E+03                     | -3.167bE+01         |
| <u></u>                          | 4 57075+C3                              | 3.4760E+02                    | 8.82156+02                  | -7.043JE-01           | 7.5078E+03                              | -2.5726E+01         |
| - ا ر ر •                        | 4.1489E+D3                              | = <b>3</b> ,7∩89E+02          | 1.28316+03                  | 6.00395E-02           | 7.1069E+03                              | <b>-1.</b> 9690E+01 |
| 1605.                            | 3 <b>.</b> 74646+0                      | 50+35226 <b>-</b> 4-          | 1.6458E+03                  | 6.8813E~∩1            | 6.74426+03                              | -1.42925401         |
| 7557.                            | 3.1438E+03                              | -7.0124E+02                   | 1.943354U3                  | 4.7387E-N1            | 6.4267E+03                              | •9.8313E+00         |
| 5°.5°.                           | 2.5162E+J3                              | -5,34525+02                   | 60+32+22 *2                 | 9.3267E-01            | 6.1558E+J3                              | -P.4411E+00         |
| e • • · ·                        | 1.83776+03                              | -3,277hE+02                   | 2.42216+03                  | 7.26595-01            | 5.9679E+03                              | -4.0446400          |
| 5: [ <b>(</b> •                  | 1.301105+13                             | -1.75456+02                   | 2.5643543                   | 10-36528.4            | 5,8257E+03                              | -2.4145E+00         |
| 1.<br>1.<br>1.<br>1.<br>1.<br>1. | 4.4042E+32                              | -8.7764E+01                   | 2.64595+U3                  | 2.7534E-D1            | 5.7241E+n3                              | -1.2905E+DO         |
| 6476.                            | <b>₽.</b> 96439E+72                     | 10+36662***                   | 2.74974703                  | 1.09555-01            | 5,650JE+03                              | -4.6472E-01         |
| 1. *n* a                         | 5.45535+02                              | -1.9J25E+N1                   | 2.7459E+03                  | -1.390 <i>3</i> t-02  | 5.594][+03                              | 1。3012F-01          |
| 1 . 1. 4. 4                      | 4.4421440                               | -2.54245403                   | 2.8414F+03                  | -1.07926-01           | 5 <b>.</b> 5487E+93                     | 6.39526-01          |
| 1.1 5:5                          | + <b>-</b> 5 5 5 4 <del>2</del> 7 5 5 7 | 1.73164+01                    | ₽.8834€+03                  | -1.83935-01           | 5.5064E+P3                              | 1.112+E+00          |
| +, ~ 2 • 1                       | 5.01165+32                              | 4.514HE+A1                    | 2.4257£+03                  | -2.44445-1]           | 5.4633E+03                              | 1.61445+00          |
| 1,54.1                           | 5.455432                                | 8, 2יו+7E+Pl                  | 2.97706+13                  | -3.0570E-01           | 5.413CE+03                              | 2.21406+00          |
| 1.4.7.4                          | 8°13246+72                              | 1.16355+UZ                    | 3.0414E+03                  | -3.54616-01           | 5.349(E+U3                              | 3.0117E+00          |
| 1.5.10                           | 1.09785+03                              | 1.31512+02                    | 3.1254E+U3                  | - 1 • 4 7 4 4 E - 0 I | 5.26316+n3                              | t_09845+ng          |
| 1.7.5                            | 1.5234573                               | 9.8152E+()1                   | <b>∃</b> .гч5,⊑+1] <b>3</b> | 10-0-5-8-1-           | 5.1.+5.+03                              | 5.57125+00          |
| e- ( • ]                         | 60+1+621+2                              | 10+30560*5-                   | 80+700T+8                   | -6.4324E+01           | 4.9741t+03                              | 7.5373E+UO          |
| 5 y y z = 1                      | 2.81525493                              | 10,53555700                   | 3.53505+03                  | -1.22755400           | 4.25405403                              | 9.9537E+AG          |
| 1.5-1                            | H = + 1 + 0 - 1 - 1                     | -4.53125+02                   |                             | -2.23~56+00           | EC+33635.4                              | 1.77285+01          |
| ∩ ± ⊂ • •                        | E0+35+EF.€                              | -3.98256+02                   | 4.25%96+03                  | -3.7454E+00           | 4.1311E+03                              | 1.5585E+01          |
| STEP NUMBE                       | r<br>ar                                 |                               |                             |                       |   |                     |
| 1.46e U                          | 3.94705403                              | 91435136-E <del>-</del>       | EC+38852°+                  | 1.47945401            | FU+311E1 +                              |                     |
|                                  |   |                               |                             |                       |   |                     |
| 2                                | E0+378+6.5                              |                               |                             |                       |   |                     |
|                                  | 2 . 4 15 6F + 0 3                       | 7.84726401                    |                             |                       |   |                     |
|                                  | 1.71805+04                              |                               |                             |                       |   |                     |
| و ن <sup>ع</sup> د               | F L1 + 1 + F 4 2 = 1                    | 1.12225                       | 5.37056+03                  | 0                     | 3.0145E+03                              | -1.95656-01         |
| . 5 : - 5 .<br>2 5               | 4 54235402                              | 1.34926+12                    | 5.41436403                  | 1.25476+00            | 2,9257E+03                              | -1.35016-01         |
| 451.5*                           | 7.45495+32                              | 9.J24JE+D1                    | 5.5359E+U3                  | 3.7641E-4)1           | 2.8541E+03                              | -7.080×E-02         |
| • 6407                           | 50+2+8+5+02                             | 5.3575E+01                    | 5.5935E+03                  |                       | 2,79645403                              | -6.92225-04         |
| 2592.                            | 5.475254.2                              | 2.37576+01                    | 5.6434£+03                  | -8.9218E-01           | 2.7454E+03                              | 7.66285-02          |
| σ<br>,                           | 5.74755+72                              | 3.01446-01                    | 5.69275+03                  | -1.43446+00           | 2.69805+93                              | 1.6551E-01          |
| · · · · · ·                      | 6.54+7F+U2                              | -2.32024.401                  | 5.74426403                  | -2.01446+90           | 2.54545+13                              | 2.7247E-01          |
| 1.511                            | 8.14526+02                              | 10+40868-51                   | 5.8054E+U3                  | -6.20196+00           | 2.54345423                              | 4.N?3E-01           |
| 1.1712                           | 1.05855+03                              | -I.2292E+02                   | 5 . 3H5 4E + U 3            | -3.52HU1+00           | 2.50425+03                              | 5.7551E-01          |
| f [ + ] • [                      |   |                               | 5.441×13.55                 | 14 - 7 7 5 5 E + 2 0  | E D + : 2 8 FE • 2                      | 10-3+E12-2          |
| •                                | L. 45446 403                            |                               | 5.13775133                  | -h.4745c+1JU          | 5.543455.54<br>5                        |                     |
|                                  |   |                               |                             |                       | 50+ 040.2                               | 1.13236+UU          |
|                                  |   |                               |                             | -1.2536E+1)L          | 1.81645403                              | 1. 2014245          |
|                                  |   |                               |                             |                       | L.515/5+03                              | 6.3414E-01          |
|                                  | *                                       |                               | 7.61055+03<br>7.65.55       |                       | 1.17445+13<br>5.5255535                 |                     |
|                                  |   |                               |                             |                       |   |                     |
| 1 · · · · ·                      |   | 2011/1/2/2/11<br>2011/11/2/20 | /.4/5htt+U3<br>co.aunof.co  |                       | 5 · · · · · · · · · · · · · · · · · · · | 10-3+6+5 • + •      |
| 7. J . L . T                     | 1. 11 [UCTU                             | C * 3 C 9 T C 1 C 1           | 504317155.8                 | 714 35832 * 51        | L.15*4C-LU                              | -T., 110C-13        |

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| STRSLT 2 |  | STRSLT Z |  | STRSLT 2 |  |
|----------|--|----------|--|----------|--|
| STRST 2  | 1.28066404<br>1.358066404<br>1.352546404<br>1.45546404<br>1.458176404<br>1.557176404<br>1.554104<br>1.564104 | STRST 2  | L - 22536+0<br>L - 28536<br>L - 295276<br>L - 4136<br>L - 4136   | STRST 2  | 1.18096404<br>1.28206404<br>1.28306404<br>1.233426404<br>1.33426404<br>1.33526404<br>1.43526404  |
| STRSL 2  | 4 * 1 8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9  | STRSL 2  | + + + + + +  | STRSL 2  | <ul> <li>W</li> <li>H</li> <li>H</li></ul>  |
| STRSLT 1 |  | STRSLT 1 | -1.07344402<br>1.11546402<br>1.11546402<br>1.11546402<br>1.202364402<br>0.22244402<br>0.22244402<br>1.25716402<br>1.25716402<br>1.3554602<br>1.3554602<br>0.3356402  | STRSLT 1 | 0<br>-2.150%E+02<br>2.152%E+02<br>0.<br>-2.1989t+02<br>2.2110E+02<br>0.<br>0.<br>-2.2587E+02<br>2.2587E+02<br>0.   |
| STRST 1  |  | STRST 1  | $ \begin{array}{c} 1 & 3 & 3 & 5 & 3 & 5 & 3 & 5 & 3 & 5 & \mathbf$  | JIRST 1  | 2. 4445<br>3. 4445<br>3. 34446<br>5. 34446<br>5. 34446<br>5. 4715<br>5. 4725<br>5. |
| STRSL i  |  | STRSL 1  | $\begin{array}{c} \mathbf{x} & \mathbf{w} & \mathbf{w} & \mathbf{v} & \mathbf{w} & \mathbf{w} & \mathbf{v} & \mathbf{w} & \mathbf{w} & \mathbf{v} & \mathbf{v} & \mathbf{w} & \mathbf{w} & \mathbf{v} & \mathbf{v} & \mathbf{w} & \mathbf{w} & \mathbf{v} & \mathbf{v} & \mathbf{v} & \mathbf{w} & \mathbf{w} & \mathbf{v} & $ | STRSL 1  | 1. 4 5 2 5 2 2 2 4 5 2 4 5 2 4 5 2 4 5 2 4 5 2 4 2 3 2 4 5 5 5 3 3 2 4 5 4 2 4 2 4 2 4 2 4 2 4 2 4 2 4 2 4   |
| LAYER    | ,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,<br>,                  | LAYER    | ー ミ キ ち み ? み つ ー ユ ー ー ー ー<br>ー ミ キ ち み ァ み つ ー ミ ラ ナ ー ー   | LAYER    | ー こ <b>ち せ ら ら ち オ り じ ー う</b> う ナ う う う て け け い う う う う う う う う う う う う う う う う う   |
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|   | STRSL 2<br>3.40506+04<br>3.44686+04<br>3.44866+04<br>3.453046+04<br>3.53046+04<br>3.53046+04<br>3.51386+04   | 87785<br>87785<br>83.33115404<br>34.33115404<br>34.33395755404<br>35.4265404<br>36.4265404<br>36.4261404<br>37.42616404<br>37.42616404   | 8TKSL 2<br>3.27666404<br>3.27666404<br>3.376404<br>3.3726404<br>3.32366404    |
| 0.<br>3.7604E+52<br>3.75246+02<br>0.<br>3.7282E+02<br>3.7201E+02<br>0.<br>1.02  | STRSLT 1<br>0<br>++.0+53E+02<br>+.0340E+02<br>0.<br>0.<br>+.0138E+02<br>0.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1  | STRSLT 1<br>STRSLT 1<br>0<br>++.18466402<br>+.18036402<br>0.1676402<br>0.1676402<br>+.16296402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1676402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.1777402<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.17774002<br>0.177740   | STRSLT 1<br>0.<br>4.2720E+02<br>4.2693E+02<br>0.                              |
| 6.84176402<br>1.88402<br>1.88406403<br>6.88406403<br>6.8356403<br>1.8858402<br>1.8558402<br>1.8558403<br>1.8558403<br>1.8558403 | STKST 1<br>8.0312E+02<br>2.1495E+03<br>2.1495E+03<br>7.4438E+03<br>7.4438E+03<br>7.4239E+02<br>7.8388E+03<br>7.8388E+03<br>7.8388E+03<br>7.8388E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.72886E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7288E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.7486E+03<br>7.748656E+03<br>7.74865656E+03<br>7.74865656565656565656565656565656565656 | <ul> <li>S. 11</li> <li>S. 12</li> &lt;</ul>   | STKS1 1<br>9.12826+02<br>2.47256+03<br>2.47296+03<br>2.47296+03<br>4.04526+03 |
| 3.44226404<br>1.74656404<br>1.73926404<br>1.73926404<br>3.339436404<br>3.339436404<br>1.71216404<br>1.70976404<br>1.70976404    | $\begin{array}{c} 81785 \\ 4 & 01776 \\ 2 & 03778 \\ 2 & 03776 \\ 4 & 01706 \\ 2 & 03736 \\ 4 & 03736 \\ 4 & 03736 \\ 4 & 04766 \\ 4 & 0466 \\ 4 & 0466 \\ 4 & 0466 \\ 4 & 0664 \\ 1 & 4 & 2 & 2 \\ 2 & 8 & 046 \\ 1 & 4 & 2 & 2 \\ 2 & 8 & 0486 \\ 1 & 4 & 046 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 5 & 6 \\ 1 & 4 & 0 & 4 \\ 1 & 4 & 0 & 0 \\ 1 & 4 & $  | STRSC<br>STRSC<br>STRSC<br>STRSC<br>STRSC<br>2.202261404<br>2.202261404<br>4.31346404<br>4.31346404<br>4.31346404<br>4.226406404<br>4.226406404<br>4.226406404<br>4.226406404<br>4.1280204<br>2.1280204<br>2.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.12804<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.1280204<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128004<br>4.128 | STRSL 1<br>+.5721++0+<br>2.3251E+04<br>2.3251E+04<br>2.3251E+04               |
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|       | 10            | 2.29555+114   | 2.447(E+113           | 4.2504E+02   |              |                         |          |
|       | 11            | 2°418144  | 2.4435E+N3            | <b>-4</b> ,2476E+U2  |              |                         |          |
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|       | nu i          | 2.40555404  | 2,5573E+D3            | <b>-4.</b> 32676432  | 3.24545404   | 9.9373E+U3              | •        |
|       | ב רד          | 10+0+0+0+0+   | 2.5553E+03            | 4.3275E~02   | 3.25376+04   | 9.96302403              | •        |
|       | r u           | 4 • /1 /11 /11 / • • •  |                       | •  | 3.26215+04   | 9.9886E+03              | •        |
|       | n u           | 7.710104<br>0.00005400  |                       | -W   | 3, 2,7050+04 | 1.0014E+04              |          |
|       | - n           |   | CUTUUCTU C            |  | 3.6/835+U4   | L.UU+UE+U+              | •        |
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|       | 13            | 4.6801E+U4  | 9.3411E+02            | .0   |              |                         |          |
|       | + 1           | 2.381hE+04  | 2.5326E+03            | 4.3116E+UZ   |              |                         |          |
|       | 15            | 2.3795E+04  | 2,5305E+03            | -4.3102E+02  |              |                         |          |
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| ×     | LAYER         | STRSL 1   | STRST L               | STRSLT 1   | STRSL 2      | STRST 2                 | STRSLT S |
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|       | ~             | 5-+30,9+-2  | 2.61.12E+D3           | 4.3651E+02   |              |                         |          |
|       | <b>c</b> n (  | +0+350E8 <b>*</b> +   | 9.63826+02            | • 0  |              |                         |          |
|       | т -           |   | 9.6350E+02            | 0 <b>.</b>   |              |                         |          |
|       |               | 1.4100/04<br>1.450000   | C. 50875113           | 4.36355+U2<br>-4.3525402   |              |                         |          |
|       | 1 -           |   |                       |  |              |                         |          |
|       | 1             | 4.82285+04  | 9.6221E+02            | •  |              |                         |          |
|       | + 1           | 2.4554E+U4  | E0+35504°2            | 4.3612E+02   |              |                         |          |
|       | 1 Z           | 2.45465+04  | 2°60446403            | -4.3607E+02  |              |                         |          |
|       | ЧT            | 4.81795+1)4   | 9.61256+02            | •0   |              |                         |          |
|       |               |   |                       |  |              |                         |          |
|       |               |   |                       |  |              |                         |          |
| ×     | LAYER         | STRSL 1   | STRST 1               | STRSLT 1   | STRSL 2      | STRST 2                 | STRSLT S |

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| 0400        | ٦              | 4.92866+04  | 9.8304E+02  | • 0                 | 3.1860E+04 | 9.7555E+03             | 0.       |
|-------------|----------------|---|---|---------------------|------------|------------------------|----------|
|             | <b>س</b> ا     | 2.5101E+04  | E0+3EP24.5  | -4.3967E+02         | 3.1851E+0+ | 9.7529E+03             | .0       |
|             | m.:            | 2.51025+04  | 2.45946+03  | 4.3968E+O2          | 3.1843E+04 | 9.7503E+03             | •0       |
|             | <b>+</b> ι     |   | 9.8317E+C2  | •0                  | 3.1835E+04 | 9,7477E+03             | •0       |
|             | <b>л</b> .     | + - + 35575 * +   | 9.83215+02  | •0                  | 3.1826E+04 | 9.7452E+D3             | •        |
|             | ا م            | <pre>&lt; .510%E+0+</pre>                               | <b>2.</b> 6598E+03  | -4.3969E+02         | 3.1818E+04 | 4.7426E+D3             | •0       |
|             | r~ 1           | 2.5107E+04  | E0+35454.5  | 4.3969E+02          |            |                        | :        |
|             | 89             | 4.4301E+04  | 9,8334E402  | •••                 |            |                        |          |
|             | σ              | +0+36364  | 9.8333E+02  | •0                  |            |                        |          |
|             | 01             | 2-5110E+04  | 2.6602E+A3  | 4.3971E+02          |            |                        |          |
|             |                | 2.5111E+04  | En+3En44.5  | -4.3472E+02         |            |                        |          |
|             | U n<br>-1 -    | **********  | 9.8351E+02  | • 0                 |            |                        |          |
|             | n 3            |   | 4 8 4 5 5 F + 5 F |                     |            |                        |          |
|             |                | 6.41717140404<br>1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1. |   |                     |            |                        |          |
|             | ۲ .<br>۲ -     | C. 51 [HE+U+  | 2.5507E+J3  | -4.3474E+U2         |            |                        |          |
|             | а <b>т</b> .   | + - + + +   | 9.8358E+N2  | •0                  |            |                        |          |
|             |                |   |   |                     |            |                        |          |
|             |                |   |   |                     |            |                        |          |
| ×           | LAYER          | STRSL 1   | STRST 1   | STRSLT 1            | STRSL 2    | STRST 2                | STRSLT   |
|             |                |   |   |                     | I          |                        | 1        |
| ۲ C F U • . | -1 N           |   | 9,9705E+02<br>2 60565403  | U.<br>              | 3.1681E+04 | 9.7009E+03             | • •      |
|             |                | 2 5430F104  | C.*34250.*03<br>C.C.4646403   | 10+U/FT+T+          | 3.154UE+04 |                        | •        |
|             | • +            | 5,0475+04   | 9.9800F+02  | 1. TOUCTUC          | 3.15575404 | 4.6/340404<br>0.007403 | <b>.</b> |
|             | S              | 5.0053E+04  | 9.98325+02  |                     | 3.15156+04 | 9 6500F403             | •<br>•   |
|             | ı<br>L         | 2.5503E+U4  | 5,64886+03  |                     | 3.14745404 | 50+3464,8              | • c      |
|             | ~              | 2.5512E+04  | 2, h996 + U3  | 4 .4222E+02         |            |                        | •        |
|             | 80             | 5.0111E+J4  | 9.9927E+02  | 0.                  |            |                        |          |
|             | σ              | 5.01276+14  | 9.4459E+n2  | 0.                  |            |                        |          |
|             |                | 2.534E+04   | 2.70205+03  | 5C+39E2+°+          |            |                        |          |
|             |                | C.30456454  | <pre>c. 2028E+03</pre>  | -+.+2%lE+02         |            |                        |          |
|             | 4 m            |   | EU+35000 1  |                     |            |                        |          |
|             | ) <del> </del> | 2.55696+04  | 2.7057F+03  |                     |            |                        |          |
|             | 7 2            | 2.55775+04  | 2.7050E+03  |                     |            |                        |          |
|             | 15             | +0+30520*5  | 1.0018E+03  | • 0                 |            |                        |          |
|             |                |   |   |                     |            |                        |          |
|             |                |   |   |                     |            |                        |          |
| ×           | LAYER          | STRSL 1   | STRST 1   | STRSLT 1            | STRSL 2    | STRST 2                | STRSLT   |
| .1465       | 1              | 5.0657E+U4  | 1.0101E+03  | 0.                  | 3.1520E+0+ | 9.6515E+D3             | •0       |
|             | N              | 2.5812E+34  | 2.7290F+03  | <b>-4</b> .4405c+02 | 3.1448E+04 | 9.6244£+03             | •        |
|             | m              | 2 • 5824.E + 0 +  | 2.2304E+03  | 4.4413E+02          | 3.1376E+04 | 9.4073E+03             | •0       |
|             | ÷ I            | 5.07406404  | 1.0116E+03  | • 0                 | 3.1304E+04 | 9.5852c+03             | •0       |
|             | <b>ر</b> م     | 5.07588434  | 1. 1122FHD3   | 0.<br>              | 3.1231E+0+ | 9.5631E+03             | • •      |
|             |                |   |   |                     | 9.115UC+U+ | 4.541154U3             | •        |
|             | · æ            | 5.025555404   | L.0138E+03  | 0.                  |            |                        |          |
|             | σ              | 5.08785+04  | 1.n1436+03  | •                   |            |                        |          |
|             | 10             | + - + - +   | 2.740115403   | 4 4 4 70 5 4 0 5    |            |                        |          |
|             | 11             | 2.59395+04  | 2.7413E+N3  | -4.79E+02           |            |                        |          |
|             | ນ ຕ<br>- 1     | 5.0461E+04<br>6 00005+00                                | 1.01600+03  |                     |            |                        |          |
|             |                |   | 50135470°T  | u.<br>4 46046403    |            |                        |          |
|             | - L'<br>-      |   |   |                     |            | -                      |          |
|             | 4 ~<br>1       | 5 1 1 2 1 E + 1 H                                       |   |                     |            |                        |          |
|             |                |   | 1   | • 2                 |            |                        |          |

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| 5.1353E+J   | 2 ELOT +(   | S I L   | SIRSLI 1<br>U.  | STRSL 2<br>3.1356€+04  | SIRST Z<br>9.6012E+03   | ЗТЯЗL° 2<br>0. |
|---|---|---|---|--|---|----------------|
| C & C C C S + C + D C + A C 2 + A + D C + A C 2 + A + D C + A C 2 + A + A + A + A + A + A + A + A + A + |   | 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| 4, 46176 + 02<br>4, 46246 + 02<br>0, 4666 + 02<br>4, 4602 + 66<br>4, 4705 + 66<br>4, 4705 + 60<br>0, 4, 715 + 60<br>0, 715 + 705  | 3.12516+04<br>3.11466+04<br>3.10426+04<br>3.09376+04<br>3.08336+04   | 2.256416+03<br>2.542016+03<br>2.542016+03<br>2.542016+03<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.456403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.557003<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.450403<br>2.45040000000000000000000000000000000000  |                |
| TR3L  | 1 ST4   | 1 1 S   | STRSLT 1  | STRSL 2  | STRST 2   | STRSLT 2       |
|   | 4 + + - 2 0 2 + 0 - + 0 - + 0 - + + - + + + + + + |   | 0,<br>4,48656402<br>0,48786402<br>0,498786402<br>4,4491666402<br>4,4491666402<br>0,498646402<br>0,498646402<br>0,498646402<br>0,498646402<br>0,44902<br>0,44902<br>0,44902<br>0,28032<br>4,4902<br>0,28032<br>4,402<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2803<br>0,2800 | 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| а. 64<br>а. 64<br>а. 43<br>а. 43<br>а. 4<br>а. 4<br>а |                |
| TRSL  | т втв   | 1 18  | STRGLT 1  | STRSL 2  | STRST 2   | STRSLT 2       |
| Мараларарана   Мараларарарана   Мараларарарана   Мараларарарана   Мараларарарана   Нараларарарана       | + + + + + + + + + + + + + + + + +                 | mmm mmm mmm mmm mmm mmm mmmm mmmm mmmmm mmmm  | 0<br>+.5175E+02<br>+.519nE+02<br>0.<br>1.5233E+02<br>1.5233E+02<br>1.5233E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5247E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02<br>1.5277E+02   | 3.07356404<br>3.07356404<br>3.0735616404<br>3.05416404<br>3.01505404<br>3.01505404<br>3.01556404   | 9.4713E+03<br>4.4115E+03<br>9.3515E+03<br>9.2515E+03<br>9.2319E+03<br>9.2319E+03<br>9.1724F+03<br>9.1724F+03  | <b></b>        |

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|   | STRSLT 2       | •••••  | STRSLT 2 |   | STRSLT 2 | •••••••<br>••••••  |
|---|----------------|--|----------|---|----------|--|
|   | STRST 2        | 4,3754£+03<br>4,24242+03<br>4,2402+03<br>4,21255+03<br>4,0465+03<br>4,0465+03<br>8,9585+03<br>8,95855+03   | STRST 2  | 9.2418E+03<br>9.1312E+03<br>9.0204E+03<br>8.7959E+03<br>8.5484E+03<br>8.6484E+03<br>8.6484E+03  | STRST 2  | 9.0510E+03<br>8.9012E+03<br>8.7515E+03<br>8.6017E+03<br>8.6017E+03<br>8.9519E+03<br>8.9029E+03   |
|   | STRSL 2        | 3.005<br>3.005<br>3.0087<br>4.047<br>4.047<br>4.047<br>4.047<br>4.047<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.04<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.044<br>4.0444<br>4.0444<br>4.0444<br>4.0444<br>4.0444<br>4.0444<br>4.04444<br>4.04444<br>4.044 | STR3L 2  | м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м<br>м  | STRSL 2  | 2.95596+04<br>2.90706+04<br>2.90706+04<br>2.85416+04<br>2.85036+04<br>2.75036+04<br>2.75036+04   |
| -4.5364E+02<br>0.                       | STRSLT 1       | 0.<br>4.55905+02<br>4.55955+02<br>0.<br>4.55955+02<br>0.<br>4.55055+02<br>4.5505<br>1.5505<br>1.5505<br>1.5505<br>1.5505<br>1.5505<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.4502<br>1.   | STRSLT 1 | 0.<br>+. 5117E+02<br>+. 6117E+02<br>0.<br>+. 6117E+02<br>+. 617UE+02<br>+. 5240E+02<br>0.<br>1.<br>+. 5310E+02<br>+. 6328E+02<br>0.<br>+. 6328E+02<br>0.<br>1.<br>+. 6328E+02<br>0.<br>0.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1  | 3TK3LT 1 | 0.<br>-4.6748E+0?<br>4.5771E+0?<br>0.<br>-4.68834E+0?<br>-4.68853E+0?<br>1.<br>-4.6932E+0?<br>1.   |
| 2.4440E+03<br>1.0770E+03                | <b>STRST 1</b> | $\begin{array}{c} 1\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\ 2\\$   | STRST 1  | 4 F - F - F - F - F - F - F - F - F - F   | STRST 1  | 1.18366403<br>1.167066403<br>2.17706403<br>1.14466403<br>1.14466403<br>1.14766403<br>2.14766403<br>2.14766403<br>2.14766403<br>2.14766403<br>2.14766403<br>3.24766403<br>2.14766403<br>3.24766403<br>2.170766403<br>3.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.2707366403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.270766403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.27076666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666403<br>2.2707666400000000000000000000000000000000 |
| 2. ~ 50 5 E + 0 +<br>5. + 0 7 0 E + 1 + | STRBL 1        | $ \begin{array}{c} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 &$   | STRSL 1  | 0         0 | STRSL 1  | 5.9437<br>6.02437<br>6.02437<br>6.02437<br>6.02437<br>6.0247<br>6.0277<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.0477<br>7.04777<br>7.04777<br>7.04777<br>7.04777<br>7.04777<br>7.047777<br>7.047777<br>7.047777<br>7.0477777777777777777777777777777777777   |
| e ی<br>ب ب                              | Larey          | - ๚ ๚ ๛ ๚ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛ ๛  | LAYER    | ┥╜╙┇╖┰┍╔┱╛┥╜╙┇╖╹<br>┙┚╙┇╖┰┍╔┱╛┥ <b>╜╙</b> ┇╖╹   | LAYEr    |  |
|   | ¥              | 1.5516   | ×<br>G   | تر<br>ج<br>ج<br>1-47  | ×        | 1.7442   |

|               | - c • + v s  | 3、78126.04<br>5、781246.04<br>5、77524.04<br>3、72225.14<br>3、10227.04<br>3、10327.04<br>3、10321.04<br>5、10551.04  | a. 210.05<br>1. 20555403<br>1. 20555403<br>1. 20555403<br>1. 2055403<br>1. 2055403<br>2. 2124603<br>1. 2135403<br>1. 2135403<br>1. 2135403   | -4.6954E+02<br>0.<br>1.4.7072E+U2<br>-4.7044E+02<br>0.   |  |  |   |
|---------------|--|--|--|--|--|--|---|
| ×             | LATER  | STRSL 1  | STRST 1  | STRBLT 1   | STRSL 2  | STRST 2  | STRSLT  |
| и<br>и<br>с   |  | $ \begin{array}{c} \mathbf{v} = \mathbf{w} = \mathbf{v} = \mathbf$ | $\begin{array}{c} 1 \\ 2 \\ $ | 0<br>++.7491E+02<br>+.7527E+02<br>+.753E+02<br>-+.753E+02<br>-+.752E+02<br>-+.7512E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>-+.7812E+02<br>7812E+02<br>7812E+02<br>7812E+02<br>7812E+02<br>7812E+02<br>7812E+02<br>7812E+02<br>7772E+02<br>7772E+02<br>7772E+02<br>7772E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>7777E+02<br>77777E+02<br>77777E+02<br>77777E+02<br>77777777777777777777777777777777777  | 2.8671E+04<br>2.8024E+04<br>2.7378E+04<br>2.5731E+04<br>2.5731E+04<br>2.5741E+04<br>2.5441E+04<br>2.5441E+04 | 8.7990E+03<br>8.5810E+03<br>8.3830E+03<br>8.1850E+03<br>8.1850E+03<br>7.9901E+03<br>7.7901E+03<br>7.7901E+03   | <b></b>   |
| L b X         | 」<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス<br>ス | $\begin{array}{c} \mathbf{S} \mathbf{S} \mathbf{S} \mathbf{S} \mathbf{S} \mathbf{S} \mathbf{S} S$  | $\begin{array}{c} 1 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\$   | STRSLT 1<br>STRSLT 1<br>0.<br>4.8318E402<br>4.8318E402<br>0.<br>1.88548E402<br>0.<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.88548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.89548E402<br>1.895486488648<br>1.8954888888648<br>1.8954886488648888888888888888888 | STRS<br>STRS<br>STRS<br>STRS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>SS<br>S    | STRST 2<br>STRST 2<br>STRST 2<br>S 1 L 3 C F + D 3<br>2 2 L 1 3 C F + D 3<br>7 5 8 5 F + D 3<br>7 4 4 U 5 8 F F + D 3<br>7 4 4 U 5 8 F F + D 3<br>7 4 4 U 5 8 F + D 3<br>7 4 4 U 5 8 F + D 3<br>7 4 4 4 0 3<br>7 4 4 4 6 7<br>7 4 4 4 0 3<br>7 4 4 4 6 7<br>7 4 4 4 6 7<br>7 4 7 7<br>7 4 7 7<br>7 7 7 7 7<br>7 7 7 7 7<br>7 7 7 7 7 7<br>7 | STRSLT<br>00.00.00.00.00.00.00.00.00.00.00.00.00. |
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|   | 2 4 5 7 8 7 8 7 8 7 8 7 8 7 8 7 8 7 8 7 8 7  | STRSL 2<br>3.72796+04<br>3.75596+04<br>3.725986+04<br>3.72356+04 |
|---|--|--|
| +.0337E+02<br>0<br><br><br>3.993.E+02<br>3.993.E+02<br>3.952EE+02<br>-3.9202E+02<br>-3.9202E+02<br>-3.9202E+02<br>0.  | STRSLT 1<br>STRSLT 1<br>4.255566402<br>4.255566402<br>4.255566402<br>4.22566402<br>4.22566402<br>4.22566402<br>4.22566402<br>4.225066402<br>4.22756402<br>4.27556402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.217566402<br>4.21756646646666666666666666666666666666666 | STRSLT 1<br>0.<br>4.294886402<br>4.29466402                      |
| <ul> <li>2.17406403</li> <li>2.13406403</li> <li>2.134364403</li> <li>2.12456403</li> <li>2.12456403</li> <li>2.12646403</li> <li>2.55606403</li> <li>2.55606444</li> <li>2.556064444</li> <li>2.55664444</li> <li>2.556644444</li> <li>2.556644444</li> <li>2.55664444444</li> <li>2.556444444</li> <li>2.55644444444</li> <li>2.5564444444</li> <li>2.5564444444444</li> <li>2.556444444444444444444</li> <li>2.55644444444444444444444444444444444444</li></ul>  | <ul> <li>N.8.8.0.0.8.8.0.8.8.0.0.8.8.0.0.8.8.0.0.8.8.0.0.8.8.0.0.8.8.0.0.8.8.0.0.8</li></ul>   | STRST 1<br>9.27766603<br>2.51266403<br>2.51266403<br>2.51596403  |
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| 0.<br>-4.2323£+42<br>4.2782£+02<br>0.  | -+ - 2+ 5+ 5+ 02<br>-+ - 2+ 1+ 5+ 02<br>0 - + + 35 + 102<br>-+ + 2+ 85 5 + 02<br>-+ - 24 85 5 + 02<br>-+ - 23 14 5 + 02<br>-+ - 23 14 5 + 02<br>+ - 23 14 5 + 02<br>+ - 23 14 5 + 02<br>-+ - 23 14 5 + 02<br>+ - 24 5 + 02 | 0.<br>0.<br>-+.2138E+02<br>-+.2194E+02<br>0.<br>+.1946+02<br>-+.1946+02<br>0.<br>14.1794E+02<br>0.<br>+.1733E+02<br>0.  |
|--|--|---|
| <b>9.1866E+02</b><br>2.4394€+03<br>2.44335€+03<br>9.1184€+03<br>9.1184€+02<br>4.09555+03 |  | 8. 84526402<br>2. 34916402<br>2. 34916402<br>2. 34916403<br>2. 34916403<br>8. 75417403<br>8. 75417403<br>8. 75517403<br>8. 75514103<br>8. 75514103<br>8. 5754103<br>8. 574416403<br>8. 574266403<br>8. 574266403<br>8. 57126403 |
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などで記載

| 1 20    | STRST 1<br>9.4560E+02<br>2.5501E+03  | STRSL 1 STRST 1<br>+.7343E+U4 9.4550E+02<br>2.4094E+U4 2.5501E+03   |
|---------|--|---|
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|  | <b>3</b> TR3 | П П П П<br>П П П П<br>П П П П<br>П П П П<br>П П П П  | STRS       | 1.01136<br>1.01346<br>1.01556<br>1.01555<br>1.01555  |
|  | STRSL 2      | 3 4 4 4 10 5 4 10 5 4 10 5 4 10 5 4 10 5 5 4 10 5 5 4 10 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5   | STRSL 2    |  |
| 0.<br>4.2561E+02<br>-4.2533E+02<br>0.                              | STR3LT 1     | - + + 0<br>- + - 355 + 4 C + 0<br>   | STRSLT 1   | 0<br>++.3719£+02<br>+.3711£+02<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.<br>1.   |
| 9.0370E+D2<br>2.4530E+D3<br>2.4492E+D3<br>8.9922E+D2<br>8.9922E+D2 | STRBT 1      | C N N C C N  | STRST 1    | <b>2.</b> 52<br><b>2.</b> 52<br><b>2.</b> 52<br><b>2.</b> 51<br><b>2.</b> 51<br><b>2</b> |
| +.52335+0+<br>2.30129€+0+<br>2.22256+0+<br>1.4256+0+<br>1.4031€+0+ | STRSL 1      | <pre> + * * * * * * * * * * * * * * * * * * *</pre>  | STRSL 1    | 4<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2<br>2   |
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| -++ 35,7E+02<br>0<br>++ 35,92E+02<br>++ 35,95E+02<br>++ 35,4E+02<br>++ 35,4E+02<br>++ 35,7E+02<br>++ 35,7E+02<br>++ 34,98E+02<br>++ 34,98E+02 ++ 34,98E+02<br>++ 34,98E+02 ++ 34,98E+02 ++ 34,98E+02 ++  | STRSLT 1<br>STRSLT 1<br>STRSLT 1<br>STRSLT 1<br>STRSLT 1<br>STRSLT 1<br>STRSLT 2<br>STRSLT 2<br>STRSLT 1<br>STRSLT 1<br>STRSLT 2<br>STRSLT 2<br>STR | STR3LT 1 |
| <ul> <li></li></ul>  | <ul> <li>8. No service service</li></ul>  | 1 13X12  |
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|  | $\begin{array}{cccccccccccccccccccccccccccccccccccc$   | 9TRSLT 1<br>9TRSLT 1<br>9TRSLT 1<br>9<br>4.4044E+02<br>4.4073E+02<br>4.4073E+02<br>4.4072E+02<br>4.4109E+02<br>4.4119E+02<br>4.4119E+02<br>4.4119E+02<br>4.4119E+02<br>4.41135E+02<br>4.41157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E+02<br>1.4157E |
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STRSLT 2

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| STRST 2 STRSLT 2 | $\begin{array}{c} 8 \\ 3 \\ 5 \\ 5 \\ 5 \\ 7 \\ 7 \\ 7 \\ 7 \\ 7 \\ 7 \\ 7$  | STRST 2 STRSLT 2 | 8.42826403 0.<br>8.30746403 0.<br>8.18756433 0.<br>8.18756403 0.   |  |            |  |  |   |   |  |  |
|------------------|--|------------------|--|--|------------|--|--|---|---|--|--|
| STRSL 2          | 4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4<br>4   | STRSL 2          | 2.752557+04<br>2.71326+04<br>2.57336+04<br>2.57336+04              | - )<br>                                |            |  |  |   |   |  |  |
| STRSLI 1         | 1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1<br>1   | STRSLT 1         | 0<br>-+-+66944+02<br>+-+6964+02<br>0.                              | 0.<br>-4,4,2,4,6,4,0,2                 | 4.4803E+02 |  | <pre>4 4 90 36 4 4 80 36 4 4 80 36 4 4 80 36 4 4 0 2 0 2 4 4 4 4</pre> | 4 4 8 0 3 6 4 7 0 2 3 6 4 8 0 3 6 4 8 0 3 6 4 8 0 3 6 4 9 0 2 5 6 4 0 2 5 6 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 6 6 7 6 7 6 6 7 7 6 7 6 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 | 4 4 4 8 0 3 4 4 4 6 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5   | <pre>4 4 90 36 4 4 90 36 4 4 90 36 4 90 36 4 90 36 4 90 36 4 90 2</pre>     | 4 4 4 8 0 3<br>0 0 4 4 4 8 0 3<br>0 0 4 4 4 8 8 0 3<br>0 6 6 4 0 2<br>0 4 4 9 8 8 4 4 0 2<br>0 4 4 9 8 7 4 0 2<br>0 4 4 9 8 7 4 0 2<br>0 4 4 9 8 7 4 0 2<br>0 4 7 9 8 7 7 1 0 2<br>0 5 6 7 1 3 6 7 1 0 2<br>0 5 6 7 1 3 6 7 1 0 2<br>0 5 6 7 1 3 6 7 1 0 2<br>0 5 7 1 3 6 7 1 0 2<br>0 5 7 1 3 6 7 1 0 2<br>0 5 7 1 3 6 7 1 0 2<br>0 5 7 1 3 6 7 1 0 2<br>0 5 7 1 1 0 5 7 1 0  |
| STRST 1          | $\begin{array}{c} \mathbf{v} \circ \mathbf{v} \\ \mathbf{v} & \mathbf{v} \\ \mathbf{v} \\ \mathbf{v} & \mathbf{v} \\ $ | STRST 1          | 1,02575403<br>2,77196403<br>2,77546403<br>1,03116403               | 1.03296+03<br>2.73496+03               | 5°-3446-03 | 0,74446+03<br>1,03825+03<br>1,04036+03<br>2,80806+03 | 74446403<br>1.013824403<br>1.0144034403<br>2.478095403<br>2.478096403<br>2.81255403<br>1.044556403   | <ul> <li>7444640</li> <li>14464640</li> <li>140342440</li> <li>140342440</li> <li>1403640</li> <li>1403640</li></ul>   | <ul> <li>V. V. 44</li> <li>V. U. 44</li> <li>V. U. 44</li> <li>V. V. 44</li> <li>V. 44</li> <li>V.</li></ul> | <ul> <li>7444640</li> <li>1444640</li> <li>144644640</li> <li>144640</li> <li></li></ul> | <ul> <li>Q. 2444 F + 6 + 0</li> <li>L. 0442 F + 6 0</li> <li>L. 0440 - 4 + 6 + 0</li> <li>R. 2440 - 4 + 10</li> <li>L. 0470 - 4 + 0</li> <li>R. 2440 - 4 + 0</li> <li>R. 24</li></ul>   |
| 5TRSL 1          | $ \begin{array}{c} \mathbf{x} \mathbf{v} \mathbf{v} \mathbf{v} \mathbf{v} \mathbf{v} \mathbf{v} \mathbf{v} v$  | STRSL 1          | 5.1+56E+U+<br>2.6250E+1+<br>2.5295E+1+<br>2.5295E+0+<br>5.1729E+0+ | 5.18276+0+<br>2.54356+0+<br>2.54356+04 |            | 5.20436+34<br>5.20436+34<br>5.7184E+04<br>2.75726+04 | 0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0   | 0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.  | 0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0<br>0.0  |  | 2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>200<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2000<br>2 |
| т., <b>т</b> .,  |  | LAYER            | -1 V m +   | ጥደሥ                                    | a          | 8 P 0  | 8 0 0 1 N  | 86040m+<br>4444-  |   | <b>8 8 0 1 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 </b>  |  |
| -                |  | ×                | l - 1 - 1  |  |            |  |  |   |   |  |  |

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|  | 81837 2<br>3.9603E+03<br>7.6600E+03<br>7.5838E+03<br>7.5838E+03  | STKST 2<br>3.3072E+03<br>7.1387E+03<br>5.9697E+03<br>6.8841E+03   |
|--|--|---|
|  | 2 3 1 8 2 3 3 4 8 7 8 2 5 3 4 3 7 8 2 5 5 3 4 3 7 4 5 7 4 5 7 4 5 4 5 4 5 4        | STRSL 2<br>STRSL 2<br>2.38666404<br>2.33146404<br>2.23046404<br>2.24826404  |
| 0.<br>4.5193E+02<br>4.5218E+02<br>0.<br>4.5228E+02<br>0.<br>1.52846<br>1.4.55246<br>1.4.55246<br>1.4.55246<br>1.4.55246<br>1.4.55246<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.552<br>1.4.5525<br>1.4.552<br>1.4.5525<br>1.4.5525<br>1.4.5   | STRSLT 1<br>STRSLT 1<br>4. 4943ELT 1<br>4. 4943EL 02<br>4. 5044602<br>4. 518846402<br>4. 518846402<br>4. 518846402<br>4. 518846402<br>4. 518886402<br>4. 5188866666<br>4. 5188866666<br>4. 5188866666<br>5. 5188866666<br>5. 51888666666<br>5. 5188866666666<br>5. 5188866666666<br>5. 51888666666666<br>5. 51888666666666666666<br>5. 518886666666666666666666666666666666666   | STRSLT 1<br>0.<br>-4.5208E+02<br>4.5255E+02<br>0.   |
| L.06146403<br>2.85561403<br>2.855505403<br>1.05866403<br>1.05866403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.88446403<br>2.9846403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.984246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.994246403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99426403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.99466403<br>2.994664003<br>2.994664000000000000000000000000000000000  | <ul> <li>NO NALLANA NALANA NALANA</li></ul> | STAST 1<br>L. n604E403<br>Z. 8645E403<br>Z. 87245E403<br>Z. 8724E403<br>L. 1704E403<br>L. 17242E403<br>L. 4724E403<br>Z. 8480E403 |
| 5.3275<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175<br>7.7175 | Ν          | STASL 1<br>S. 32485404<br>S. 32485404<br>č. 72915404<br>č. 72875404<br>5. 32257404<br>5. 32257404                                 |
| ー ご m ナ y ェ ^ y チ G i ー Q<br>Q Q Q Q Q Q Q Q Q Q Q Q Q Q Q Q Q Q   | 4<br>7<br>7<br>7<br>8<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9<br>9  | ፈ<br>ሥ<br>ጠ ጣጣሠት የኦፕ<br>አ   |
|  | × +<br><br><br><br><br>G-59  | × ۍ<br>•<br>•   |

STRSLT 2

STRSLT 2

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|     | c                   | 1.0850F+03         | 5.44786+04          | -1           |
|-----|---------------------|--------------------|---------------------|--------------|
| 316 | STHSLT 1            | 3TRST 1            | STRSL I             | LaYčk        |
|     |                     |                    |                     |              |
|     | •0                  | 1.1639£+03         | 5.85086+04          | ℃<br>FI      |
|     | -4.6471E+02         | 3.1058E+03         | 2.97026+04          |              |
|     | 4 <b>.</b> 6431E+02 | 3.0475E+U3         | 2.9016E+04          | IF.          |
|     | •0                  | 1.15+0E+D3         | 5.800AE+14          | ታ<br>ጊ       |
|     | .0                  | 1.15076+03         | 5.78315414          | с<br>С       |
|     | -4.6309E+02         | 3.0727E+03         | 2.93576+04          | 57           |
|     | 4.6267E+U2          | 3 0 0 4 4 F + 0 3  | 2.92715+114         | 2 b          |
|     | <b>ם</b>            | 1.1407E+03         | 5.7321E+04          | 50           |
|     | • 0                 | E0+342ET"T         | 5.7151E+U+          | ±<br>∿       |
|     | 4.61436+02          | 3.039hE+03         | 40+12166 <b>-</b> 5 | 6 S          |
|     | -4.6101E+02         | <b>a.</b> nala6+na | 2.84256404          | د ک          |
|     | •0                  | 1.1274E+03         | 5.6642E+A4          | Ч, Ч         |
|     | •0                  | 1.1241E+03         | 5 . 64 726 +114     | -<br>v       |
|     | 4.5473E+02          | 3,0064E+03         | 2.86676+1)4         | ъT           |
|     | -4.5430E+02         | 2,94805+13         | 2.8581E+U+          | 19           |
|     | •0                  | 1.11416+03         | 5.5463E+04          | 17           |
|     | <b>.</b> 0          | 1.1108E+03         | 5.5/946+04          | ١b           |
|     | -4.5800E+02         | E0+J1F7P.5         | 2.83225+04          | 15           |
|     | <b>+.5755E+0</b> 2  | 2.464HF+03         | 2.82365+04          | + -1         |
|     | <b>.</b> D          | 1.1008E+03         | 5.52846+04          | r.<br>-1     |
|     | 0.                  | 1.0475E+03         | 5.5115E+U4          | ru<br>T      |
|     | <b>-+</b> .5423E+02 | E0+34969.5         | 2.29226+0+          | 11           |
|     | 4.5577E+02          | <b>2.4</b> 34164   | 2.7H41E+14          | 1 1          |
|     | <b>.</b> D          | 1.0875E+03         | 5.4535545454        | <del>,</del> |
|     | •0                  | 1.1442E+03         | 5.4436E+14          | y            |
|     |                     |                    |                     |              |

| ×      | LaYčk          | 5723L I            | <b>3TRST 1</b>          | STHSLT 1               | STRSL 2    | STRST 2             |
|--------|----------------|--------------------|-------------------------|------------------------|------------|---------------------|
| 1.4465 | 1              | 5.44785+04         | 1.0850E+03              | • 0                    | 2.1023E+04 | 6.4371E+03          |
|        | പ              | 2.78595+1)4        | <b>2.4</b> 283E+03      | <b>-+.</b> 5559£+02    | 2.05066+04 | 6.22885+0           |
|        | m              | 2.74776+04         | 2,9347F+D3              | 4.5521E+02             | 1-99896+04 |                     |
|        | Ŧ              | 5.51786+04         | 1.09876+03              | •0                     |            | 6 - 0 + 0 + C + C + |
|        | ۍ<br>س         | 5.54116+14         | 1.1033E+U.              | •                      |            |                     |
|        | a.             | +0+3-663-5         | 60+31+25 <b>~2</b>      | <pre>** 580+E+05</pre> |            |                     |
|        | ~              | 2.84526+04         | <b>2</b> .9855E+D3      | 4.53645+02             |            |                     |
|        | <b>n</b> 0     | 5.hillE+0+         | 1.1170E+D3              | •0                     |            |                     |
|        | σ              | 5.6345E+04         | 1.1216E+03              | •0                     |            |                     |
|        | 10             | 2.88075+04         | 3.0198E+03              | 4 604 E+U2             |            |                     |
|        | 11             | P.842554           | 3,03125+03              | -4.61005+02            |            |                     |
|        | ۶ <b>۲</b>     | 5.7U45E+04         | 1.1353E+ú3              |                        |            |                     |
|        | 14             | 5.7278E+A+         | 1.1344E+03              | 0.                     |            |                     |
|        | + 1            | 2.92826+134        | 3°01545403              | 4 627(E+02             |            |                     |
|        | 15             | 2.940112+04        | 3.074HE+03              | <b>-4.5328E+02</b>     |            |                     |
|        | ЧT             | 5.297HF+U+         | 1.1535€+03              | 0                      |            |                     |
|        | 17             | 5.8211E+0+         | 1.15815+03              | 0.                     |            |                     |
|        | ۶T             | 2.97556+114        | 3.1104F+03              | -+-5+8+6+02            |            |                     |
|        | ЪТ             | 2.9474E+24         | 3.12236+03              | 4 <b>b5</b> 49E+02     |            |                     |
|        | 202            | 5.84116+04         | 1.1718E+03              | .0                     |            |                     |
|        | 2 I            | 5.91446+14         | 1.1763E+03              | 0.                     |            |                     |
|        | <del>ر</del> ک | 3.02305+34         | 3.15h4£+n3              | -4.6710E+U2            |            |                     |
|        | с Э            | 40+36+E0.F         | 3.1676E+03              | 4.6762E+02             |            |                     |
|        | <b>າ</b><br>ເບ | 5.9844E+114        | 1.1400E+D3              | •0                     |            |                     |
|        | 52             | 6.0078E+n+         | 1.1446E+03              |                        |            |                     |
|        | 42             | 3 ° N 204 E + 114  | 3.24]554(13             | 4.69195+02             |            |                     |
|        | <b>ر</b> ج     | 9 . 1963 3E + 14   | EU+48clr*F              |                        |            |                     |
|        | ж<br>V         | 6.07775+04         | E0+~2802*T              | 0.                     |            |                     |
|        | 5 V            | <b>h</b> .1011E+04 | 1.21285+D3              | 0.                     |            |                     |
|        | с F            | 3.11746+34         | <b>J</b> • + 5 7E + N 3 | 4.7121E+U2             |            |                     |

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STRSLT 2

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|                          | STRSLT 2 | •                        | •••         | •                        |             |            |             |                          |             |             |            |                           |            |            |                           |                          |                   |             |            |                          |                  |              |                          |                  |  |            |   | STRSLT 2 | •          | •           | •0                  | •0                                       |             |             |                           |                |             |                                  |              |                     |            |
|--------------------------|----------|--------------------------|-------------|--------------------------|-------------|------------|-------------|--------------------------|-------------|-------------|------------|---------------------------|------------|------------|---------------------------|--------------------------|-------------------|-------------|------------|--------------------------|------------------|--------------|--------------------------|------------------|--|------------|---|----------|------------|-------------|---------------------|--|-------------|-------------|---------------------------|----------------|-------------|----------------------------------|--------------|---------------------|------------|
|                          | STRST 2  | 5.3504E+03<br>E 2607E+03 | 5.1509E+03  | 5.1004E+03               |             |            |             |                          |             |             |            |                           |            |            |                           |                          |                   |             |            |                          |                  |              |                          |                  |  |            |   | STRST 2  | EO+36680"+ | 4.0856E+03  | 4 ° 08786+03        | 4.089UE+03                               |             |             |                           |                |             |                                  |              |                     |            |
|                          | 31KSL 2  | 1.7473E+04<br>1.2144F+04 | 1.6822E+04  | 1.bb57E+04               |             |            |             |                          |             |             |            |                           |            |            |                           |                          |                   |             |            |                          |                  |              |                          |                  |  |            |   | STRSL 2  | 1.3335E+04 | 1.3343E+04  | 1.33505+04          | 1.3354E+04                               |             |             |                           |                |             |                                  |              |                     |            |
| -4.71726+02<br>0.        | STRSLT 1 | 0.<br>-4.59396402        | 1- 6018E+02 | •                        | -4.6251E+02 | 4.6327E+02 | •           | U.<br>4.65516+02         | -4-66246+02 | 0.          | •0         | 4.6838E+02<br>#4 Lanvelin | 0.         | • 0        | -+.7113E+02               | 4.71805+02               | • •               | -4.73776+02 | 4°3447E+05 | <b>.</b>                 | u.<br>*.7629E+02 | -4.76906+02  | • •                      | יט.<br>בחינוכר א | 2013010/°1                             |            |   | STRSLT 1 | с.         | -4.6331E+UZ | 4 - 6 4 2 9 E + U 2 | •  | -+.6715E+02 | 4 .68085+02 |                           | 4 . 7079E+02   | -4.7167E+U2 | • =                              | ₩.7423E+02   | -4.7505E+uc         | <b>U</b> . |
| E0+34222°E               | STRST 1  | 1.11216+03<br>2.00026+03 | E0+39510*E  | 1.13066+03<br>1 13686+03 | 3.0518E+03  | 3.U?72E+D3 | 1.1553E+U3  | 1.12975403<br>3.12975403 | 3.1385E+U3  | 1.1799E+D3  | 1.1841E+03 | 3.18446+D3<br>2 19436403  | L.2045F+D3 | 1.2107E+D3 | EU+3+5+6 m                | 4.60155+13<br>60431540 1 | 1.2352E+03        | 3.31,41E+03 | 3.3213E+03 | ].7536E+03<br>1 36895403 | 3,3566E+03       | . 3.3317c+03 | 1.2782E+U3               | CUTICES.1        |  | 1.3025F+A3 | - | STRST 1  | 1.1415E+U3 | 3.0783E+n3  | E0+3+840 E          | 1.1257E+03                               | 3.1585E+A3  | 3.17856+03  | 1.70746+133<br>5.0546+133 | E 0 + JE + D 3 | 3.25426+D3  | 1. 2.400047403<br>1. 2.201047103 | 3, 31,785+03 | E0+39765.E          | 1.2621c+03 |
| 3.1297E+N4<br>6.1709E+04 | STRSL 1  | 5。58636+04<br>2。86046+14 | 2.87645+14  | 5.58086+0+<br>5.21236+04 | 2.4244E+04  | 2.9407E+U4 | 5.80685 +14 | 2.9885E+0+               | 3.0045E+04  | 4.4328F.+0+ | 5.96436+04 | 3.U525E+04<br>3.D585F404  | h.0583E+U+ | 6.040JE+04 | 3.1165E+04<br>2 .2255.200 | 3.14736+04<br>F 1848F454 | 6.2163E+j+        | 3.1×05E+04  | 3.1965E+U4 | 5.310КЕ+U4<br>г ачазетич |                  | 3.250555+04  | Ь.458£+194<br>С.86005.00 |                  | 10110000000000000000000000000000000000 | 5626F+U+   |   | STRSI 1  | 5.7364E+U4 | 2.94155+194 | 2.9625E+34          | 5. X 5 1 1 F + 0 F<br>5. 4 1 1 2 F + 0 F | 3.02546+04  | 3.04536+04  | 6.024364194<br>6.0563646  | 3.1091E+04     | +U+311E1*:  | 5, 18955+04                      | 3.19246+04   | 4 <b>.</b> 2138F+05 | 6.35446+14 |
| LE<br>GL                 | LAYER    | 0                        | m           | <del></del>              | ם. ו        | ~ :        | x 0         | 10                       | 11          | τc          | EI         | + u<br>                   | 1 P        | 17         | H 1                       | r =<br>- 1               | ) - <b>1</b><br>2 | 22          | ក :<br>ក : | + Մ<br>Ն Ն               | ם.<br>ריני       | <b>ر</b> ح   | 8 0<br>N N               | 5                | T.F                                    | 15         |   | LAYER    |            | പ           | m :                 | + u                                      | · T         | r .         | x 7                       | υT             | 11          | ი.<br>                           | , <b>+</b>   | 15                  | Ļ          |
|                          | ×        | 1.5315                   |             |                          |             |            |             |                          |             |             |            |                           |            |            |                           |                          |                   |             |            |                          |                  |              |                          |                  |  |            |   | ×        | 1.11.5     |             |                     |  |             |             |                           |                |             |                                  |              |                     |            |

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| +103       -+.67386+02       9.15776+03       2.80416+03         +03      98546+02       9.57366+03       2.917866+03         +03       0      975466+03       2.917866+03         +03       0      975466+03       2.917866+03         +03       0      975266+03       2.917866+03         +03       0      975266+03       2.917866+03         +03       0      975266+03       2.917866+03         +03       0      9717266+03       2.917866+03         +03       0      9717266+03       2.917866+03         +03       0      9717266+03       2.917866+03         +03       0      971266+02       9.15766+02         +03       0      971266+02       9.15766+02         +03       0      971266+02       9.1566+02         +03       0      971266+02       9.15666         +03       0      971266+02       9.15666         +03       0      971266+02       9.15666         +03       0      97126666       9.15666         +03       0      97126666       9.15666         +03       0      97126666       <  | STRST 2<br>STRST 2<br>1.28486403<br>1.455403  | STRSL 2<br>4.17b1E+03   | STKSLT 1<br>0.<br>  | STRST 1<br>1.2114E+03<br>3.25215+03                                  | STRSL 1<br>9443E+1)4<br>1342E+1)4   | <br><br>   | ۲. ۲<br>۲<br>۲<br>۳. ۲<br>۳. 1<br>۳. 1                                    |
|---|---|---|---|--|---|--|---|
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|   |   |   |   |  |   |  |   |
|   |   |   | 4.8620E+02<br>-4.86866+02   | ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ<br>ມີ | 3.6314<br>3.6513  | 3.5272E+1)4 3.6319<br>3.5487E+1)4 3.6513<br>7.1134E+104 1.2001 | 30 3.52226+04 3.6319<br>31 3.54826+04 3.6513<br>42 3.013464.04 1.5601     |
|   |   |   | •   | 60+3   | 1.3582  | 6.8448E+04 1.3582<br>6.8400E+04 1.3582                         | 24 5.84886+04 1.3582<br>29 5.840055+04 1.3582                             |
| +0 0 1  |   |   | 0.<br>4.23450402<br>14.24450402   | n m m<br>5 + + +   | 2373<br>- 667<br>- 137<br>- 1<br>- 137<br>- 1 |  |   |
| +03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03  |   |   | •   | m ()<br>+ ()<br>- ()   | 12 4 7 8 • 1<br>12 4 7 8 • 1  | 6.634556434 F. F. BARA966. 202526435                           | 25 5.2525554154 1.32575   |
| +03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03<br>+03  |   |   | 50+3h218+1  | m () +   | ່ມ<br>ເມີ   |  |   |
| <ul> <li>4. 81246-102</li> <li>4. 81246-102&lt;</li></ul> |   |   | 0.<br>-4 80555403   | е<br>С<br>С<br>+<br>+  | 1.3028  | 8.55045+04 I.90225<br>3.35036+04 I.43625                       | 22000 T T T T T T T T T T T T T T T T T                                   |
| 103       -4.80555+02         +03       -4.812465+02         +03       0.  |   |   | •   | E U +  | 1.29425<br>1.2025   | 6.5192E+04 1.2942E<br>F. ELOMETADE 1.2013E                     | 2-1 6-51928+04 1.204297<br>2-1 5 55055404 - 1 202255                      |
| <pre>+n3 +n3 +n3 +n3 +n3 +n3 +n3 +n3 +n3 +n3</pre>  |   |   |   |  |   |  |   |
|   |   |   | 0.  | E 0+   | 1.2702E   | 5. 3955E+0+ 1. 2702E   | 12 5.3456E404 1.2702E   |

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|        | r            | 3.16596+04                          | 3.29215+03   | 4°2005+05                 | 5.1737E+03 | 1.5842E+03 | .0   |
|--------|--------------|-------------------------------------|--|---------------------------|------------|------------|------|
|        | <b></b>      | 6.2811E+U+                          | 1.2+785+03   | <b>.</b> 0                | 5.4215E+03 | 1.6601E+03 | .0   |
|        | л.           |                                     | 1.25007+03   | 0                         |            |            |      |
|        | 0 11         | 3.65034C+04<br>2 2024C+04           |  | -4.7681E+02<br>. J000E.02 |            |            |      |
|        | . <u>a</u>   | 53025404                            | 50+35435°'   |                           |            |            |      |
|        | œ            | 6.54255404                          | 1.3084E+03 ·   |                           |            |            |      |
|        | 61           | 3. 3H7 4= +11+                      | 3.50086+03   | 4.81436+02                |            |            |      |
|        | 1.1          | 3°41416+11+                         | <b>3.5</b> 404€+03   | -4.8753E+02               |            |            |      |
|        | 15           | 6.77945+1)4                         | ].3447E+O3   | 0.                        |            |            |      |
|        | E T          | 6 <b>8</b> 416E+1)4                 | 1.3558E+03   | •0                        |            |            |      |
|        | + L          | 3.5139E+0+                          | 3.61H9E+03   | 4.8567E+02                |            |            |      |
|        | ۲ ۲<br>۲     | 3.54556404                          | 3.643E403  | -4.8668E+02               |            |            |      |
|        | с г<br>      | 7.02355+04<br>7.00035+04            | 1.3930E+03   | •                         |            |            |      |
|        | . H          |                                     |  |                           |            |            |      |
|        | د ه<br>۱     |                                     | 1.1375515<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.15755<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.157555<br>1.1575555<br>1.1575555<br>1.1575555555555 |                           |            |            |      |
|        |              | 2.2776404                           |  | 4 • 40440450              |            |            |      |
|        | ไว           | 7.33985404                          | 1.45326403   |                           |            |            |      |
|        | 22           | 3.76646+0+                          | 3.85265403   | -4-43156+02               |            |            |      |
|        | εJ           | 3.74855+0+                          | 3.4316E+03   | 4.9400E+02                |            |            |      |
|        | +<br>വ       | 7.52475+04                          | 1.48425+03   | 0                         |            |            |      |
|        | 2 S          | 7.58646+04                          | 1.50)3E+03   | <b>.</b> u                |            |            |      |
|        | 47           | 3 <b>.</b> 84345.404                | ЕО+Э£ячь°Е   | 4 . 9 4 3 5 4 0 5         |            |            |      |
|        | <b>د</b> ک   | 3.925nE+04                          | E0+314PP.E   | -4.9721E+02               |            |            |      |
|        | 28           | 7.7758ć+J4                          | 1.5372E+03   | 0.                        |            |            |      |
|        | 59           | 7.83805+04                          | 1.5492E+03   | .0                        |            |            |      |
|        | ÛE           | 4°0198E+14                          | 4.0%32E+03   | 4,94446402                |            |            |      |
|        | TF           | 4°0214E+04                          | 4.1118E+03   | -5.0J14E+02               |            |            |      |
|        | ЪЕ           | 8.U245E+U4                          | 1.5850F+03   | .0                        |            |            |      |
|        |              |                                     |  |                           |            |            |      |
| ×      | LAYER        | STRSL 1                             | STAST 1  | STRSLT 1                  | STRSL 2    | STRST 2    | STRS |
| 0518.1 | 4            | 6.3318E+U4                          | L.25775+03   | -=-                       | • 0        | 0.         | .0   |
|        | ~ <i>i</i>   | 3.25930+04                          | 3.33046+03   | -4.76725+02               | ••         | ••         | •0   |
|        | n :          | +0+3FSF7 F                          | 3 4 14 3E + 03   | 4.7808E+02                | •0         | ••         | ••   |
|        | <del>.</del> |                                     | 1.2341E+03   | •                         | • 0        | •          | •    |
|        | ר ע          | 6.6154C+04<br>2 40335404            | L.J.C.T.+0.3   | .u.<br>                   |            |            |      |
|        | ; ~          |                                     |  | 20130100 H                |            |            |      |
|        | - 20         | 6.8481E+04                          | 1.35416+03   |                           |            |            |      |
|        | Ŧ            | 6.8490E+04                          | 1.36746+03   |                           |            |            |      |
|        | υT           | 3 5 4 7 4 6 4 1 4                   | 3.4510F4D3   | 4.46705+02                |            |            |      |
|        | 11           | 3.58346+134                         | 3.6×34E+03   | -4.8782E+02               |            |            |      |
|        | 12           | 7.1117E+;J4                         | 1.4091E+03   | .0                        |            |            |      |
|        | t T          | 7.182hE+:J+                         | 1.4228E+n3   | <b>.</b> .                |            |            |      |
|        | + 1          | 3.64145404                          | 3.7842E+03   | 4.9102E+02                |            |            |      |
|        | 15           | 3 . 7 2 7 4 5 4 0 4                 | 3.8163E+03   | -4-0503E+05               |            |            |      |
|        | r :          | 7.34535+04                          | 1.45345403   | • • •                     |            |            |      |
|        | ~ -          | 2.4662c+04                          | 1.4276E+03   | 0.                        |            |            |      |
|        | 13           | 40+1+198 P                          | 1.474712.5<br>55474712.5   | 10+4F7+7 <b>*</b> +1      |            |            |      |
|        |              | 10+04T/04F                          |  | - 4534E+DC                |            |            |      |
|        |              |                                     | L.SLADE+03   | • =                       |            |            |      |
|        | 1 NC         | 3.97445+114                         | 4.04456+03   | -4.98445+02               |            |            |      |
|        | 5 3          | 4 。//1545+U4                        | 4°0741E+03   | 4 . 9 9 2 3 E + D 2       |            |            |      |
|        | + '<br>"     | 7.9h2+E-114                         | 1.57315+03   | <b>л.</b>                 |            |            |      |
|        | ית<br>רי     | 8.03337+4650.8                      | 1.5467E+03   | 0.<br>                    |            |            |      |
|        | t            | * - + - + + - + - + - + - + - + - + | 4.1/00011.1  | 1.017747104               |            |            |      |

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|  | ·  |   |      |  |   |  |   |
|  |  |   |      |  | ti.                                       |  |   |
| + 5.02+1E+02<br>0.<br>5.0455E+02<br>1.<br>0.<br>1.<br>0.<br>0522E+02<br>1.                                     |  |   |      |  |   |  |   |
| 4.2.1896+02<br>1.5.2.256+73<br>1.5.4.116+03<br>4.30576+03<br>4.33786+03<br>4.33786+03<br>1.53176+03            |  |   |      |  |   |  |   |
| 4.15538+34<br>8.215538+34<br>8.21557+04<br>8.21547+04<br>4.25531540<br>4.357316+04<br>8.55317+04<br>8.53317+04 |  |   |      |  |   |  |   |
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# APPENDIX H

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# NONLINEAR FORMULA PREDICTIONS OF COMPLEX JOINT FAILURE LOADS/BEHAVIOR

## APPENDIX H.1

n.

FAILURE PREDICTION RESULTS IN COMPLEX JOINTS 501 COMPLEX DOUBLE LAP JOINT TITANIUM TO BORON, LSHE ADHESIVE

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0 មា ទា ເກ 1 1 0 ULTIVATE LOAD PREDICTION BASED ON ADPESIVE - MX STRESS, SU = 7.17E+03 ADPENIVE - MX STRESS, SU = 7.17E+03 ADPENEVO I - MX STRESS, SU = 7.17E+03, ST = 4.01E-03, SLT = 1.50E+02 ADPENEVO 2 - MX STRESS, SL = 1.35E+05 0 552**.**48 ທ ≁ SECANT S -0 -1910 11910 7950 SECANT S 134000 AD-FSIVE THISNESS = .0040 PUISONS WAITO = .40 HATEG 05000 CONSTANTS (SHEAR BTRESS-STRAIN CURVE) HATEG 05000 CONSTANTS (SHEAR BTRESS-STRAIN CURVE) HATEG 05000 CONSTANTS (SHEAR BTRESS-STRAIN CURVE) SCANT S = 3740 M VALUE = 6.318 Ð o NONLI'EAR URTHUTHUPIC ANALYSIS, DUUBLE LAP JOINT 4 0 .... .0900 .3052 . 16096000 3 3 3 -136532000 2750000 933000 0 0 -45 4 **-1** -1 .0850 Э ADMENEND NUMBER I(ORTHOTROPIC) THICKNESS THICKNESS NUMBER OF LAYERS NUMBER OF LAYERS NUMBER OSGOOD CONSTANTS SL VS. ET AD-EFERD NUMBER Z(ISOTROPIC) THICANESS POISSONS RAIIO RATERG OSGOOD CONSTANTS S VS. E JOINT LEAGTH = 1.4573 E4409 TULERANCE = .025 Maximuv ITEPATIONS = 20 Nummer of Stations = 61 Effective k = .020 4.2813E+00 4.2613E+U0 6.1671E+00 ALGEA B BE1A B H-3

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|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|----------------|-------------|--------------|--------------|-------------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|----------------|-------------|--------------|--------------|--------------|--------------|---------------|--------------|--------------|--------------|-------------|---------------------|
|              |              |              |              |              |              |              |              |              | ADHE           |             |              |              |                   |              |              |              |              |              |              |              |              | ADHE           |             |              |              |              |              |               |              |              |              |             | ADHE                |
|              |              |              |              |              |              |              |              |              | IN             |             |              |              |                   |              |              |              |              |              |              |              |              | N              |             |              |              |              |              |               |              |              |              |             | Z                   |
| 9.325-02     | 7.75E-02     | 6.50±-02     | 5.46E-92     | 4.59E-02     | Э.н7Е-02     | 3.26E-02     | 2.75E-D2     | 2.33E-02     | .ABLE = .950   |             | 2.066-01     | 1.175-01     | 8.82E-02          | 7.17E-02     | 5.946-02     | 5.075-02     | 4.326.72     | 3.63E~02     | 3.155-02     | 2.71E-02     | 2.335-02     | ASLE = .978    |             | 1.436-01     | 8.246-02     | 5.25E-02     | 5.106-02     | 4.306-02      | 3.67E-02     | 3.15E-02     | 2.73E-02     | 2.36E-02    | AHLE = <b>.</b> 989 |
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| A I LEXALION | AT ITEPALION | AT ITEVATION | AT ITERAFIUN | AT ITE ATIUN | AT IYERATION | AT ITERATION | AT ITERATION | AT IIC ATION | MALINIA STRESS | L0AJ = CA0J | AT ITERATION | AT IFE-ATION | AT ITEZATION      | AT L'EVALION | AT ITERATION | AT I'ERATION | AT ITERATION | AT ITERATION | AT ILLAATION | AT ITERATION | AT L'EZATION | MAKIAUI STRESS | LOAN = 5959 | AT I TAATION | AT ITEXATION | AT ITERATION | AT ITERATION | AT ITELEADION | AT ICERATION | AT ICEMATION | AT I'LRATION | AT IFLATIUN | MAXI WUM STRESS     |
|              |              |              |              |              |              |              |              |              |                |             |              |              |                   |              |              |              |              |              |              |              |              |                | •           |              |              |              |              |               | H-           | 5            |              |             |                     |

NO CONVERSENCE AFTER TEN ITERATIONS - TERMINATED

THE PREDICTED ULTIMATE LOAD IS 6077

458 " (8 + 1)b b = 8033

and the second second

NUMARK OF ITERATIONS = 9 , MAXIMUM ERROR = .02357

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|-------|------------|-------------|-------------|-------------|-----------------|---------------------|-------------|----------------------|-------------|-------------|--------------|--------------------|------------|---------------|-------------|-------------------|--------------------|--------------|-------------|
| NXZ   | 1.19185+04 | 1.0874E+04  | 9.8711E+D3  | 8.91666+03  | 8.01475+03      | 7.1724E+03          | 6.3482E+03  | 5.70456+03           | 5.1085E+03  | 4 6324C+03  | 4, 2742E+03  | 3.9812E+U3         | 3.6893E+U3 | 3,33415+03    | 2.85225+03  | 5 - 7 + 2 H + 0 H | 1.58855+03         | 8.31715+02   | 2.3283E-10  |
| TXW   | 1.90435-13 | -3.8053E-01 | -6.1671E-01 | -7.3254E-01 | -7.47906-01     | -6.87446-01         | -5.79166-01 | 10-385484-1 <b>-</b> | -3.0804E-01 | -1.680-5-01 | -2.4.3964.62 | 1.2244E-01         | 8.65525-71 | 3.9324r-01    | 4.3140E-01  | 5.11585-01        | 4.5573Emol         | 10+120+0.2   | 5.45705-14  |
| TXN   | 1.1542E-10 | 5.22015E+02 | 1.0233E+03  | 1.5005E+03  | 1.9515E+D3      | 2.37265+03          | 2.75985+03  | 3.10656+03           | 3.4046E+03  | 3.64256+03  | 3.6223E+03   | <b>Э.</b> 9683Е+03 | 4.1140E+03 | 4,2912E+03    | 4.5278E+03  | 4.8219E+U3        | 5.16466403         | 5.5430E+03   | 5.9589E+03  |
| SIGMA | 1.0291E+03 | 9.2337E+U2  | 7.75346402  | 5.42046+02  | 3.81305402      | 1.52305+12          | -7.65AlE+Ul | -2.7530E+02          | -3.8745E+02 | -3.5152E+02 | -1.9781E+U2  | -2.1428E+01        | 9.4042E+01 | 9.65735+01    | -6.b522E+U1 | -3.2512E+02       | <u>-5.8046£+02</u> | -7.839 iE+02 | -9.0830E+U2 |
| TAU   | 6.5693E+03 | 6.3112E+03  | 6.0294E+03  | 5.7191E+03  | 5.37346+03      | 4 <b>.</b> 9822E+03 | 4°5635+03   | 3 <b>.</b> 95635+03  | 3.3144E+03  | 2.5355E+03  | E0+316691    | 1.72225403         | 1.92076+03 | 2.5112E+03    | 3.2×08E+03  | E0+3E3E8*E        | 4.4557E+03         | 4 . 8863E+D3 | 5.260UE+03  |
| *     | 6000-6     | 5.7HG.      | •1557       | 5++2"       | 07.5 <b>7</b> . | 54                  | ^ 7 1 J •   | • 5 7: 5             | C           | . 7 4 5     | • 10<br>15   | • 11 · 11 •        | 0107       | и<br>тас<br>• | L. L+1 *    | 1.2225            | 1.3 +              | 1. 3. J.     | 1.4370      |

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#### APPENDIX H.2

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In Street

## BEHAVIOR PREDICTION FOR GIVEN LOADS IN COMPLEX JOINTS

COMPLEX DOUBLE LAP JOINT TITANIUM TO BORON, LSHE ADHESIVE 501

NONLINEAR ORTHOTROPIC ANALYSIS, DCUBLE LAP JOINT

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|       | .4 6.4000E+03 0. | 11 5.5394E+03 0.     | 11 4 7504E+03 0. | 11 4.0381E+03 0. | 11 3.42776+03 0. | 11 2.4511E+03 0.                  | 11 2.6291E+03 0. | 11 2.42676+03 0.    | 12 2.2981E+03 0. | 12 2.2121E+03 0.   | 12 2.1464E+C3 D. | 2 2.0868E+J3 D. | 11 2.0166E <sup>2</sup> J3 0. | 11 1.914, f+02 0. | 11 1.7696E+03 0. | L 1.5321E+03 0. | 1 1593E+D3 0. | 11 6.3582E+02 0. | 5 1.1642E-10 0. |
|-------|------------------|----------------------|------------------|------------------|------------------|-----------------------------------|------------------|---------------------|------------------|--------------------|------------------|-----------------|-------------------------------|-------------------|------------------|-----------------|---------------|------------------|-----------------|
| Ĩ     | 6.3665E+         | -3.0441E-            | -4 - 6820E-1     | -5.1943E-        | -+-851+E-I       | -3°1719E-                         | -2.8182E-        | -1.7134E-(          | -8.3851E-(       | - <b>]</b> .9941E- | 2.8143E-(        | 7.3030E-(       | 1.25366-1                     | 1.9032E-(         | 2.6312E-(        | 3.2058E-(       | 3.2205E-(     | 2.2097E-(        | 9.0949E         |
| TXN   | 5 8208E-11       | 4.3029E+02           | 6.2480E+02       | 1.1810E+03       | 1.4862E+03       | 1.7244E+03                        | 1.8855E+03       | 1.48666403          | 2.0510E+03       | 2.0440£+03         | 2.1265E+03       | 2.1566E+03      | 2.1917E+03                    | 2.2405E+03        | 2.3152E+03       | 2.4340E+03      | 2.6204E+03    | 2.8821E+03       | 3.20006+03      |
| SIGMA | 9°6791E+05       | 8.3612E+02           | 6.1721E+D2       | 3.2049E+02       |                  | -3.092BE+02                       | -4.2933E+02      | <b>-4.1</b> 449E+02 | -3.1857E+D2      | -1.8435E+02        | -3.4940E+01      | 9.1242E+01      | 1.8922E+02                    | 2.2912E+D2        | 1.8210E+02       | 1.5736E+01      | -2.74846+02   | -5°7566£+02      | -7.7046E+02     |
| TAU   | 5.4550E+03       | 5.0799E+03           | 4.6373E+03       | 4°0436260°+      | 3,3827E+03       | <b>6 * + 2</b> 3 3 5 <b>+ 0</b> 3 | 1.5579E+D3       | 9.7644E+02          | 6.3229E+02       | 4.4551E+02         | 3.6895E+02       | 3.8405E+02      | 4.9451E+02                    | 7.2757E+02        | 1.14246403       | 1.8295E+D3      | 2.7753E+03    | 3.6049E+03       | 4.2121E+03      |
| -     | 00000            | 1)<br>1.<br>1.<br>1. | .1520            |                  | •                | • • 0 25                          | ロンジナ・            | •5365•              | 0.019.           | 3992               | .8150            | . 89h5          | 0978.                         | 1.0545            | 1.1410           | 1.22<5          | 1.3040        | 1.3655           | 1.457O          |

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| STR9L    | ••••••••<br>••••••  | STRSL    |
| STRST 2  | 2. 1 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7  | 3TR31 2  |
| STR9L Z  | 7.111E+0+<br>7.111E+0+<br>7.111E+0+<br>7.1111E+0+<br>7.1111E+0+<br>7.1111E+0+<br>7.1111E+0+<br>7.1111E+0+<br>7.1111E+0+ | STRSL 2  |
| STRSLT 1 | ••••••••••••••••••••••••••••••••••••••  | STRSLT 1 |
| STRST 1  |   | STRST 1  |
| STRSL 1  |   | STR3L 1  |
| LAYER    |   | LAYER    |
| ×        |   | ×        |
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| ER | STR3L 1         | STRST 1       | STRSLT 1           | STRSL 2                                       | STRST 2    | STRSL |
|----|-----------------|---------------|--------------------|---|------------|-------|
|    | 7.3462E+D3      | 1.4778E+D2    | •                  | 6.1549E+04                                    | 1.8846E+04 |       |
|    | 3.7706E+03      | 4.2509E+D2    | <b>-1.1624E+02</b> | 6.1549E+04                                    | 1.88455+04 |       |
|    | 3.7946E+03      | 4.2777E+O2    | 1.1694E+02         | 5.15495+04                                    | 1 8845F404 |       |
|    | 7.4872E+03      | 1.5052E+02    | .0                 |   |            | • -   |
|    | 7.5342E+03      | 1.51556+(12   | •0                 | 6 - 1 - 4 - C - C - C - C - C - C - C - C - C |            |       |
|    | 3.8554E+D3      | 4 358 E+02    | 1.19036+02         |   |            | •     |
|    | 3.89045+03      | 4.38446+02    | -1-195591-1-       |   |            | •     |
|    | 7.6751E+03      | 1.5434E+02    | .0                 |   |            | • -   |
|    | 7.7221E+U3      | 1.5534E+02    | .0                 |   |            |       |
|    | 9 4 4 7 7 4 U 3 | 4 4 5 5 4 U D |                    |   |            | •     |

| STRSLT 2   | •••••••   | 3TRSLT 2<br>0.00.00.00.00.00.00.00.00.00.00.00.00.0  | STRSLT 0.00.00.00.00.00.00.00.00.00.00.00.00.0  |
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| STRSL 2  | 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2   | 4 t t t t t t t t t t t t t t t t t t t  | STRSL 2<br>3.80856+04<br>3.80856+04<br>3.80856+04<br>3.80856+04<br>3.80856+04<br>3.80856+04<br>3.80856+04 |
| 1.22516+02<br>0.<br>1.24586+02<br>1.25276+02<br>0.<br>STRSLT 1   | -2.0983E+02<br>-2.0983E+02<br>-2.1070E+02<br>0.<br>2.130E+02<br>-2.1417E+02<br>-2.1417E+02<br>-2.1759E+02<br>0.<br>1759E+02<br>0.<br>2.2098E+02<br>0. | STRSLT 1<br>22.7756402<br>22.78506402<br>2.81506402<br>2.81506402<br>2.81506402<br>2.8156402<br>2.84456402<br>0.87376402<br>2.84456402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.87376402<br>0.876766<br>0.876766<br>0.876766<br>0.876766<br>0.876766<br>0.876766<br>0.876766<br>0.876766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.877766<br>0.8777666<br>0.8777666<br>0.8777666<br>0.8777666<br>0.87776666<br>0.87776666<br>0.877766666<br>0.87776666666666666666666666666666666666  | STRSLT 1<br>0.<br>-3.2441E+02<br>3.2449E+02<br>0.<br>3.2666E+02<br>3.2666E+02                             |
| <pre>4.4920E+02 1.5817E+02 1.5912E+02 4.59323E+02 4.5940E+02 1.6195E+02 1.6195E+02 3.7837 1</pre>              | 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8   | <ul> <li>4.4.4.4.4.4.4.4.4.4.4.4.4.4.4.4.4.4.4.</li></ul>  | STRST 1<br>5.251871<br>5.25187<br>1.446846403<br>1.47256403<br>2.24676403<br>5.31186402<br>1.48476403     |
| 3.98626+03<br>7.86366+03<br>7.91006+03<br>4.05816+03<br>4.08206+03<br>8.05106+03<br>8.05106+03<br>8.05106+03   |   | 8 188.<br>8 189.<br>8 189.<br>8 189.<br>8 189.<br>8 189.<br>8 194.<br>8 194. | STRSL I<br>2.6518826404<br>1.338366404<br>1.34226404<br>2.64106404<br>2.64106404<br>2.54856404            |
| 11<br>12<br>15<br>15<br>15<br>15<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76<br>76 |   | ィ<br>イ<br>イ<br>ー コミヨー コー ユー ユー ユー<br>ビ<br>ロー コミー マー コー ユー ユー ユー<br>コー コー コー ユー ユー ユー ユー  | א מש+ מש<br>ד<br>א מש+ מש   |
| ×  | . 1630  | ະ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ<br>ເ   | x (<br>.5<br>0<br>•   |

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| <br>   | STRSLT 2 |  | STR3LT 2 |  | STRSLT 2<br>0.<br>0.  |
|--|----------|--|----------|--|---|
| 1.1.16626404<br>1.16626404<br>1.16626404<br>1.166276404  | STRST 2  | +     + <th>STRST 2</th> <th>8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8</th> <th>STRST 2<br/>8.2562E+03<br/>8.2562E+03</th>   | STRST 2  | 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8  | STRST 2<br>8.2562E+03<br>8.2562E+03   |
| ж в<br>9 - 0 - 4<br>1 - 0 - 4<br>2 - 1 - 1 - 1<br>2 - 1 - 1<br>2 - 1 - 1<br>2 - 1 - 1<br>2 - 1<br>21<br>2 - 1<br>2 | STRSL 2  | н по   | STRSL 2  | 000<br>000<br>000<br>000<br>000<br>000<br>000<br>000   | 51RSL 2<br>2.6963E+04<br>2.5963E+04   |
| -3.27225+02<br>0.228895+02<br>3.28895+02<br>3.29445+02<br>0.331095+02<br>3.31095+02<br>0.331535+02<br>0.31535+02   | STRSLT 1 | 0.<br>3.55226+02<br>3.55226+02<br>3.55226+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55286+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.5586+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+02<br>3.55886+0200586+0200000000000000000000000000000000000 | STRSLT 1 |  | 3TRSLT 1<br>0.<br>-3.8353E+02   |
| L, \$887<br>5. 357<br>5. 357<br>5. 357<br>5. 377<br>5. 377<br>1. 500<br>7. 500<br>7. 500<br>7. 500<br>1. 500<br>1. 500<br>1. 500<br>1. 500<br>7. 12<br>5. 47<br>8. 47<br>8. 47<br>8. 400<br>1. 500<br>7. 12<br>1. 500<br>7. 100<br>7. 100  | STRST 1  | $ \begin{array}{c} 1 2 4 5 5 1 3 4 5 5 1 3 5 5 5 1 1 3 5 5 5 5 1 1 3 5 5 5 5 1 1 3 5 5 5 5 5 1 1 5 5 5 5 5 5 5 5$  | STRST 1  | $\begin{array}{c} \mathbf{A} = $ | STKST 1<br>7.1361£+02<br>1.9657£+03   |
| 1.35756+04<br>2.671366+04<br>2.671366+04<br>2.671366+04<br>1.37316+04<br>2.370166+04<br>2.370166+04<br>1.388456+04<br>1.388956+04<br>1.388956+04<br>1.388956+04  | STRSL 1  | $\begin{array}{c} 1 & 0 & 0 & 0 \\ 1 & 5 & 6 & 3 & 0 \\ 1 & 5 & 6 & 5 & 6 & 3 \\ 1 & 5 & 6 & 5 & 6 & 5 & 6 \\ 1 & 5 & 5 & 6 & 6 & 6 & 6 \\ 1 & 5 & 5 & 6 & 6 & 0 & 6 \\ 1 & 5 & 7 & 7 & 1 & 0 & 6 \\ 1 & 5 & 7 & 7 & 1 & 0 & 6 \\ 1 & 5 & 7 & 8 & 6 & 0 & 6 \\ 1 & 5 & 1 & 1 & 5 & 0 & 6 \\ 1 & 5 & 1 & 2 & 6 & 0 & 6 \\ 1 & 5 & 1 & 2 & 6 & 0 & 6 \\ 1 & 5 & 1 & 2 & 6 & 0 & 6 \\ 1 & 5 & 1 & 5 & 0 & 6 & 0 & 6 \\ 1 & 5 & 1 & 5 & 0 & 6 & 0 & 6 \\ 1 & 5 & 1 & 5 & 0 & 6 & 0 & 6 \\ 1 & 5 & 1 & 5 & 0 & 6 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 1 & 0 & 0 & 6 \\ 1 & 5 & 0 & 0 & 0 & 0 & 6 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 \\ 1 & 5 & 0 & 0 & 0 \\ 1 & 5 & 0 &$   | STRSL 1  | L  | SIRSL 1<br>3.5653E+04<br>1.8179£+04   |
| ~~~~<br>HHHHHH   | LAYER    |  | LAYER    |  | ГАҮ<br>БА<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са<br>Са |
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|      | 4     |               |                     |                |  |                          |        |
|------|-------|---------------|---------------------|----------------|--|--------------------------|--------|
|      | n ±   |               | 1.9571E+03          | 3.8378E+02     | 2.5453E+0+                               | 8.25625+03               | •      |
|      | ۲.    | +0+0+0-0.0*0  | 7 • 1 5 4 0 E + 0 2 | • •            | 2.64635+04                               | 8.2562E+03               | •      |
|      | n     | 3.57616+04    | 7.1574E+02          | •              | 2.5963E+04                               | 8.2562E+D3               | •      |
|      | a     | 1.8234E+04    | 1.9713E+03          | 3.84195402     | 2.69636+04                               | 8.25626+03               | •      |
|      | ~     | 1.8248E+04    | 1.9727E+03          | -3.8433E+02    | 2.6963E+04                               | 8 <b>.</b> 2562E+03      | •0     |
|      | ø     | 3.58416+04    | 7.17346+02          | <b>.</b><br>C  | 2.69535+04                               | 8.2562E+03               | .0     |
|      | σ     | 3.58525+04    | 7 1 7 8 7 5 + 0 5   |                |  |                          | i c    |
|      | 10    | 1.8289E+0+    | 1.97696+03          | -3,8475F+D2    |  |                          | •      |
|      | 11    | 1.83026+04    | 1.97836+03          |                |  |                          |        |
|      | 12    | 3.5949E+04    | 7.19476+02          |                |  |                          |        |
|      | 13    | 3.59766+04    | 7.20005+02          |                |  |                          |        |
|      | 14    | 1.83435404    |                     | 2.8731F+DV     |  |                          |        |
|      | 15    | 1.83576+04    | 1.9839F+D3          |                |  |                          |        |
|      | 16    | 3.6056E+04    | 7.2160E+02          | 0.             |  |                          |        |
|      |       |               |                     | •              |  |                          |        |
|      |       |               |                     |                |  |                          |        |
|      |       |               |                     |                |  |                          |        |
| ×    | LAYER | STRSL 1       | STRST 1             | STRSLT 1       | STRSL 2                                  | STRST 2                  | STR9L  |
|      | -     | MUT 20207 C   | CUT3000C C          | c              | -0+0=ruu r                               |                          | c      |
|      | 1 N   |               | 20120100°.          | -3 000E+03     | 10+11+11+11+11+11+11+11+11+11+11+11+11+1 | /.81855+U3<br>7 0/015403 | •<br>• |
|      | u n   | 1.88286+04    | 2.03196+03          | 3.9014E402     | 2.55348404                               | 7.81855403               |        |
|      | +     | 3.6968E+04    | 7.3969E+D2          | • 0            | 2 • 2 2 3 4 U + U +                      | 7.8186E+03               |        |
|      | ſ     | 3.6981E+04    | 7.39956402          | - 0            | 2.55346+04                               | 7.81866+03               |        |
|      | . JO  | 1.88485+04    | E0+30+E0 "d         | 3,90345402     |  | EC+39010-1               |        |
|      | ~     | 1.88556404    | 5.134 hEU.4         |                |  |                          |        |
|      | - 00  | 3.7020E+04    | 7.4073E+02          | 0.             | 2.5534F+04                               | 7.81855+03               | • •    |
|      | σ     | 3.7033E+04    | 7.40446+02          |                |  | 7.818FF+13               | •      |
|      | 10    | 1.88755+04    | E0+369E0.5          | -3,5061E+02    |  |                          | •      |
|      | 11    | 1.8881E+04    | 2.0374E+03          | 3.9067E+02     |  |                          |        |
|      | 75    | 3.7073E+04    | 7.4177E+02          | •0             |  |                          |        |
|      | ET    | 3.7086E+04    | 7.4203E+02          |                |  |                          |        |
|      | * 7   | 1.8902E+04    | 2,0344E+03          | 3.9087E+02     |  |                          |        |
|      | 15    | 1.8908E+04    | 2.0401E+03          | -3.9093E+02    |  |                          |        |
|      | 16    | 3.7125E+04    | 7.4231E+D2          | ••             |  |                          |        |
|      |       |               |                     |                |  |                          |        |
| 1    |       |               |                     |                |  | p                        |        |
| ĸ    | LAYEK | SIRGL I       | STRST I             | STRGLT 1       | STRSL 2                                  | STRST 2                  | STRSL  |
| 3335 | -     | 3.7789E+04    | 7.5594E+02          | • 0            | 2.4578E+04                               | 7.52596+03               | • 0    |
|      | N     | 1.9253E+04    | 2.0752E+03          | 19.9425E+05    | 2.45785+04                               | 7.5259E+03               | •      |
|      | m :   | 1.9255E+04    | 2°0754E+03          | 3.9427E+02     | 2.4578E+04                               | 5254E+03                 | •      |
|      | •     | 3.27475+04    | 7.56135402          | •              | 2 - + 2 7 8 E + O +                      | 7.5259E+03               | •      |
|      | un i  | 3.7800E+04    | 7.5614E+02          | •              | 2.4578E+04                               | 7.52596+03               | •      |
|      | n     | 1.92505+04    | 2.0758E+03          | 3.9431E+02<br> | 2.45785+C4                               | 7.5259±+03               | •      |
|      |       |               |                     |                |  |                          | •      |
|      | 0 0   | 701 JUT07 - E |                     | - c            | r.+5/8F+0+<br>v 45,38F+0F                | 7.5254E+U3<br>7.5250F+A3 |        |
|      | 10    | 1.92556+04    | 2.0765E403          |                |  |                          | •      |
|      | 11    | 1.92586+04    | 2.0767E+03          |                |  |                          |        |
|      | 12    | 3.78226+04    | 7.5663E+02          |                |  |                          |        |
|      | ET    | 3.7825E+04    | 7.56695+02          | .0             |  |                          |        |
|      | 11    | 1.735+04      | 2.0771t+03          | 3.74445+02     |  |                          |        |
|      | 15    | 1.92746+04    | 2.0771E+53          | -3.94466+02    |  |                          |        |
|      | 16    | 3.7835E+04    | 7.568 E+U2          | • 0            |  |                          |        |

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| STRSLT 2 | 00000000  | STRSLT 2 |   | STRSLT 2 | ••••••••••••••••••••••••••••••••••••••  |
|----------|---|----------|---|----------|---|
| STRST 2  | 7. 3 304<br>304<br>304<br>304<br>304<br>304<br>304<br>304<br>304<br>304   | STRST 2  | 7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03<br>7.0498E+03  | STRST 2  | <ul> <li>4</li> <li>5</li> <li>6</li> <li>6</li> <li>7</li> <li>8</li> <li>9</li> <li>9&lt;</li></ul> |
| STRSL 2  | N       N           | STRSL 2  | 2.31870<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.31876<br>2.404<br>2.31876<br>2.404<br>2.31876<br>2.404<br>2.31876<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404<br>2.404 | STRSL 2  | <b>2</b><br><b>2</b><br><b>2</b><br><b>2</b><br><b>2</b><br><b>2</b><br><b>2</b><br><b>2</b>  |
| STRSLT 1 | -3.47326+02<br>3.47326+02<br>0.<br>3.47246+02<br>3.47246+02<br>-3.47266+02<br>-3.47266+02<br>-3.47266+02<br>3.47056+02<br>3.47056+02<br>-3.47056+02   | STRSLT 1 |   | STRSLT 1 | -++ 032556+02<br>++032566+02<br>0<br>++03166+02<br>0<br>++02906402<br>-++02<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0<br>0   |
| STRST 1  | <ul> <li>2. b 88 3 E + 0 2</li> <li>2. b 88 3 E + 0 3</li> <li>3. b 88 5 2 E + 0 3</li> <li>5. b 88 5 8 + 0 3</li> <li>5. b 88 5 8 + 0 3</li> <li>5. b 88 5 4 + 0 3</li> <li>6. b 88 5 4 + 0 3</li> <li>7. b 88 5 4 + 0 3</li> <li>8. b 88 5</li></ul> | STRST 1  | 7.80746403<br>2.139066403<br>2.139066403<br>2.139066403<br>2.139066402<br>2.13916402<br>2.13916402<br>2.13316402<br>2.13326403<br>2.13326403<br>2.13326403<br>2.13196403<br>2.13196403<br>2.13196403<br>2.13196403<br>2.13196403  | STRST 1  | <b>2.</b> 15 15 15 15 15 15 15 15 15 15 15 15 15  |
| STRSL 1  | ана и и и и и и и и и и и и и и и и и и   | STRSL 1  | $\begin{array}{c} 3 & 4 \\ 1 & 2 & 3 \\ 1 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 4 & 3 \\ 2 & 3 & 3 \\ 3 & 3 \\ 3 & 3 & $  | STRSL 1  | <ul> <li>M N N W W N N W W N N W W N N W W N N W W N N W W N N W M N N N W M N N M W N N M W N N M W N N M W N N M W N N M M N N N M M N N N M M N N N M M N N M M N N M M N N M M N N M M N N M M N N M M N N M M N N M M N N N M M N N N M M N N N M M N N N M M N N N M M N N N M N N N M N N N M M N N N M M N N N M N N N M N N N N M N</li></ul>  |
| LAYER    | - こ g + ら ら の の の う ら g + ら ら ら る - ら う う ま う ら - う う ・ う ら - う う ・ う ら - う う ・ い ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・ ・   | LAYER    | ти w то да а и и и и и и и и и и и и и и и и и  | LAYER    | ┙いちょうしょ ひらり ひょうちょうしょ  |
| بر       | 0<br>7<br>3<br>6  | ×        | υ<br>-0<br>σ<br>α   | ×        | ດ<br>ພ<br>ຕ   |

Contraind.

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|                  | ×        | LAYER  | STRSL 1   | STRST 1                                   | STRSLT 1  | STRSL 2   | STRST 2  | STRSLT                                |
|------------------|----------|--|---|---|---|---|--|---------------------------------------|
| <b>1</b> • 0 5 6 | ιn<br>σ  | - こう ナ ら ら へ g g g u u u u u u u u u u u u u u u u | <ul> <li>4.0.4.</li> <li>0.0.4.</li> <li>0.0.2.2.</li> <li>0.0.2.2.</li> <li>0.0.2.2.</li> <li>0.0.2.2.</li> <li>0.0.4.</li> <li>0.0.5.</li> &lt;</ul>      | 8   | <pre>-+.0747E+02<br/>+.0734E+02<br/>+.0734E+02<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.<br/>0.</pre>   | 2.<br>2.<br>2.<br>2.<br>2.<br>2.<br>2.<br>2.<br>2.<br>2.  | • • • • • • • • • • • • • • • • • • •  | •••••••                               |
|                  | ×        | LAYER  | STR9L 1   | STRST 1                                   | STRSLT 1  | STR9L 2   | STRST 2  | STRSLT                                |
| 11-15            |          | 4544745<br>444444<br>444444                        | <ul> <li>4</li> <li>6</li> <li>7</li> <li>8</li> <li>8</li> <li>8</li> <li>8</li> <li>9</li> <li>9&lt;</li></ul> | 8 2 2 3 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 | 0.<br>1.13546+02<br>1.13376+02<br>1.125966+02<br>1.125966+02<br>1.125966+02<br>1.125066+02<br>1.125066+02<br>1.12176402<br>1.12176402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.114964602<br>1.114964602<br>1.114964602<br>1.114964602<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496402<br>1.11496466402<br>1.11496466664<br>1.11496666666666666666666666666666666666 |   |  | ••••••••<br>•••••••                   |
|                  | ×        | LAYER  | STRSL 1   | STRST 1                                   | STRSLT 1  | STRSL 2   | SIRSI 2  | STRSLT                                |
| 2<br>2<br>•      | ሆነ<br>የህ | うちし ちらう ちらう ちっし<br>しょう                             | <ul> <li>4.</li> <li>7.</li> <li>8.</li> <li>8.</li> <li>9.</li> &lt;</ul>      | 8 8 6 6 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7   | 0<br>++.223bE+02<br>+.223bE+02<br>0<br>+.2159E+02<br>+.2159E+02<br>+.2192E+02<br>+.20b2E+02<br>.0<br>0<br>0<br>0  | 1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404<br>1. 70236404 | 2.2124<br>2.2124<br>2.22124<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2.2224<br>2 | • • • • • • • • • • • • • • • • • • • |

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|   | STR3LT 2 |   | STRSLT 2 | STRSLT 2<br>9.00.00.00.00.00.00.00.00.00.00.00.00.00  | STRSLT 2 | ••••••••••••••••••••••••••••••••••••••  |
|---|----------|---|----------|---|----------|---|
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## 13. ABSTRACT

Development of analysis methods for orthotropic adherend bonded lap joints which account for material nonlinearities at room temperature was the primary objective of the research reported herein. The use of these methods in predicting mechanical behavior, ultimate loads, and failure modes was the goal. In order to accomplish this, new analytical procedures were developed and successfully checked with discrete element techniques for single, double, and step lap adhesively bonded attachment configurations. Experimental verification of these nonlinear analyses was accomplished by the fabrication and evaluation of a variety of simple joint specimens under static monotonically increasing load. Failure loads and modes were used as the primary substantiation characteristics but the mechanical behavior of a small number of these simple joint specimens was observed at intermediate loadings and found to compare favorably with the analytically predicted behavior. Larger, more complex bonded joints were designed, fabricated, and evaluated under static monotonically increasing loads at room temperature utilizing these methods. Ultimate load, failure mode, and detailed strain behavior at any intermediate load were accurately predicted with the new analyses, as substantiated by experimental observations. These techniques were put into a computerized design/analysis program for structural application use and the program was used to generate bonded joint design allowable curves.

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| Joints                            |          |       | 1       |        |      |       |  |
| Composites                        |          |       |         |        |      |       |  |
| Nonlinear Behavior                |          |       |         |        |      |       |  |
| Design                            |          | i     |         |        |      |       |  |
| Analysis                          |          |       |         |        |      |       |  |
| Testing                           |          |       |         |        |      |       |  |
| Computer Programs (JTSDL & JTSTP) |          |       |         |        |      |       |  |
| Boron/epoxy                       |          |       |         |        |      |       |  |
| Titanium                          |          |       |         |        |      |       |  |
| Adhesion                          |          |       |         |        |      |       |  |
| Surface Preparation               |          | 1     |         |        |      |       |  |
| Polymer Processing                |          | 1 1   |         |        | ļ    | 2     |  |
| Bonding                           |          |       |         |        |      |       |  |
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