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RESEARCH ON STRUCTURAL DYNAMIC TESTING
BY IMPEDANCE METHODS. VOLUME I.
STRUCTURAL SYSTEM IDENTIFICATION FROM
MULTIPOINT EXCITATION

William G. Flannelly, et al

Kaman Aerospace Corporation

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**USAAMRDL TECHNICAL REPORT 72-63A
RESEARCH ON STRUCTURAL DYNAMIC
TESTING BY IMPEDANCE METHODS
VOLUME I
STRUCTURAL SYSTEM IDENTIFICATION FROM
MULTIPOINT EXCITATION**

By

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November 1972

EUSTIS DIRECTORATE

**U. S. ARMY AIR MOBILITY RESEARCH AND DEVELOPMENT LABORATORY
FORT EUSTIS, VIRGINIA**

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KAMAN AEROSPACE CORPORATION
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This program was conducted under Contract DAAJ02-70-C-0012 with Kaman Aerospace Corporation.

This report contains the theoretical derivation and the presentation of a methodology for system identification of structures. Computer experiments were run to verify this methodology.

The report has been reviewed by this Directorate and is considered to be technically sound. It is published for the exchange of information and the stimulation of future research.

This program was conducted under the technical management of Mr. Arthur J. Gustafson, Technology Applications Division.

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RESEARCH ON STRUCTURAL DYNAMIC
TESTING BY IMPEDANCE METHODS

Volume I
Structural System Identification From
Multipoint Excitation

Final Report

Kaman Report R-1001-1

By

William G. Flannelly
Alex Berman
Nicholas Giansante

Prepared by

Kaman Aerospace Corporation
Bloomfield, Connecticut

for

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| 13. ABSTRACT The parameters in Lagrange's equations of motion, mass, stiffness, and damping for a mathematical model having fewer degrees of freedom than the linear elastic structure it represents may be determined directly from measured mobility data and the approximate natural frequency associated with each mode. The natural frequencies are readily available from response plots. Thus, using only impedance-type test data without the use of an intuitive mathematical model, the equations of motion for the structure may be obtained -- a process referred to as system identification. In conjunction with the determination of the aforementioned parameters, the eigenvector or mode shape and generalized mass corresponding to each natural frequency are also calculated. A digital computer program was generated to numerically test the system identification theory. Computer experiments were conducted to test the sensitivity of the theory to errors in input data. | | | |

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FOREWORD

The work presented in this report was performed by Kaman Aerospace Corporation under Contract DAAJ02-70-C-0012 (Task 1F162204AA4301) for the Eustis Directorate, U. S. Army Air Mobility Research and Development Laboratory, Fort Eustis, Virginia. The program was implemented under the technical direction of Mr. Joseph H. McGarvey of the Reliability and Maintainability Division* and Mr. Arthur J. Gustafson of the Structures Division.** The report is presented in four volumes, each describing a separate phase of the basic theory of structural dynamic testing using impedance techniques.

Volume I presents the results of an analytical and numerical investigation of the practicality of system identification using fewer measurement points than there are degrees of freedom. The parameters in Lagrange's equations of motion, mass, stiffness, and damping for a mathematical model having fewer degrees of freedom than the linear elastic structure it represents may be determined directly from measured mobility data. Volume II describes the method of system identification wherein the necessary impedance data are experimentally determined by applying a force excitation at a single point on the structure. Volume III presents a method of determining the free-body dynamic responses from data obtained on a constrained structure. Volume IV describes a method of obtaining the equations for the combination of measured mobility matrices of a helicopter and its subsystems. The response of the combination of a helicopter and its subsystems is determined from data based on the experimental results of the main system and subsystems separately.

*Division name changed to Military Operations Technology Division.

**Division name changed to Technology Applications Division.

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LIST OF SYMBOLS

- [c] the damping matrix
- [d] a damping matrix; $[d] = \omega[c]$; for damping forces which are proportional to displacement
- {f} vector of external forces acting along the generalized coordinates
- $\tilde{\{f\}}$ force phasor, $\{f\} = \tilde{\{f\}}e^{i\omega t}$
- g_i the structural damping coefficient of the i-th mode
- i or j indices; imaginary operator ($i = \sqrt{-1}$)
- χ_i the generalized stiffness of the i-th mode
- [k] the stiffness matrix
- m_i the generalized mass of the i-th mode
- [m] the mass matrix
- N or n the number of degrees of freedom in the structure
- $\{\dot{y}\}$ vector of velocities of the generalized coordinates
- $\tilde{\{\dot{y}\}}$ velocity phasor, $\{\dot{y}\} = \tilde{\{\dot{y}\}}e^{i\omega t}$
- $[Y_{(\omega)}]$ matrix of mobilities at forcing frequency ω ;
 $[Y_{(\omega)}] = [\partial \tilde{y}_i / \partial f_j]_{(\omega)}$

LIST OF SYMBOLS (Continued)

- $Y_i^*(\omega)$ generalized mobility of the i -th mode at forcing frequency ω
- $\ddot{[Y]}$ matrix of acceleration mobilities
- $[\dot{z}(\omega)]$ matrix of impedances at forcing frequency ω ;
 $[z(\omega)] = [\partial \tilde{f}_i / \partial \dot{y}_j](\omega)$
- $\dot{z}_i^*(\omega)$ generalized impedance of the i -th mode at forcing frequency ω
- $\overline{\dot{z}_i^*(\omega)}$ complex conjugate of the i -th mode generalized impedance at forcing frequency ω
- $|\dot{z}_i^*(\omega)|$ absolute value of the i -th mode generalized impedance at forcing frequency ω
- $\{\gamma\}_i$ the i -th column of $[\Gamma]$; the gamma vector of the i -th mode; a left-hand eigenvector of $[k]^{-1}[m]$
- $[\Gamma]$ the left-hand eigenvectors of $[k]^{-1}[m]$; $[\phi]^{-T}$
- δ_i^j Kronecker's delta
- $\{\phi\}_i$ the modal vector of the i -th mode
- $[\Phi]$ matrix of modal vectors
- ω forcing frequency
- Ω_i the natural frequency of the i -th mode

LIST OF SYMBOLS (Continued)

SUPERSCRIPTS

- R the real part of a complex quantity
- I the imaginary part of a complex quantity
- * a generalized parameter associated with a
 particular mode
- T the transpose
- T the inverse transpose

SUBSCRIPTS

- (ω) the forcing frequency at which the quantity was
 measured or calculated
- k forcing frequency

A dot over a quantity indicates differentiation
with respect to time

BRACKETS

- [], () matrix
- [↓] diagonal matrix
- { } column or row vector

INTRODUCTION

The success of a helicopter structural design is highly dependent on the ability to predict and control the dynamic response of the fuselage and mechanical components. Conventionally, this involves the formulation of intuitively based equations of motion. Ideally, this process would reduce the physical structure to an analytical mathematical model which would predict accurately the dynamic response characteristics of the actual structure. Obviously, the creation of such an intuitive abstraction of a complicated real structure requires considerable expertise and inherently includes a high degree of uncertainty. Structural dynamic testing is required to substantiate the analytical results. The analysis is modified until successful correlation is obtained between the analytical predictions and the test results. Finally, the mathematical model can be used to incorporate changes to improve the structural integrity of the helicopter.

This report describes the theory of structural dynamic testing using impedance techniques as applied to a mathematical model having fewer degrees of freedom than the structure. Reference 1 describes the method of obtaining a model directly from test measurements for a hypothetical structure which has the same number of degrees of freedom as the mathematical model. In reality, the number of degrees of freedom of a physical structure is infinite; therefore, the usefulness of model identification, necessarily with a finite number of degrees of freedom, using impedance testing techniques depends on the ability to simulate the real structure with a small mathematical model.

The process of deriving the equations of motion from test data is referred to as system identification. The only input information required in this theory is measured mobility data and the approximate natural frequency of each mode. This information can be obtained from impedance testing of the actual structure over the frequency range of interest yielding the second order, structurally damped linear equations of motion.

System identification theories to be of any practical engineering significance must be functional with a reasonable degree of experimental error. In this report, a series of computer experiments incorporating experimental errors was documented. This report presents a modification and extension of the analysis derived in Reference 1 such that an identified model with a finite number of degrees of freedom simulates the actual structure wherein the number of degrees of freedom is infinite.

THEORY

DERIVATION

The equations of motion in matrix form of a linear system are, as shown in Reference 1,

$$[m]\{\ddot{y}\} + [c]\{\dot{y}\} + [k]\{y\} = \{f\} \quad (1)$$

Assume a steady-state solution of the form

$$\{\dot{y}\} = \{\tilde{y}\}e^{i\omega t} \text{ and } \{f\} = \{\tilde{f}\}e^{i\omega t}$$

Substitute these equations into Equation (1) to give

$$\left[([m]\omega - \frac{1}{\omega}[k])i + [c] \right] \{\tilde{y}\} = \{\tilde{f}\} \quad (2)$$

or

$$(i[z_{\omega}^I] + [z_{\omega}^R])\{\tilde{y}\} \equiv [z_{\omega}]\{\tilde{y}\} = \{\tilde{f}\}$$

where $z_{ij}(\omega)$ is defined herein as the element velocity impedance measured at ω .

The element impedance can also be expressed as

$$z_{ij}(\omega) = \partial \tilde{f}_i / \partial \tilde{y}_j$$

If Equation (2) is premultiplied by $[\phi]^{-T}[\phi]^T$ and post-multiplied by $[\phi][\phi]^{-1}$ where $[\phi]$ is the matrix of modal vectors, the result is

$$[\phi]^{-T} \left[i([\phi]^T [m] [\phi] \omega - \frac{1}{\omega} [\phi]^T [k] [\phi]) + [\phi]^T [c] [\phi] \right] [\phi]^{-1} = [z_{(\omega)}] \quad (3)$$

The diagonal generalized mass is expressed by

$$[M] = [\phi]^T [m] [\phi] \quad (4)$$

The diagonal generalized stiffness is given by

$$[K] = [\phi]^T [k] [\phi] \quad (5)$$

Assume that

$$\frac{1}{\omega} [gk] = [\phi]^T [c] [\phi] \quad (6)$$

such as would be expected from structural damping in a lightly damped structure. Substituting Equations (4), (5) and (6) into Equation (3) yields

$$[\dot{z}(\omega)] = [\phi]^{-T} \left[j(\eta\omega - \frac{k}{\omega}) + \frac{gk}{\omega} \right] [\phi]^{-1} \quad (7)$$

Define the i -th modal impedance as

$$\dot{z}_i^*(\omega) = j(\eta_i\omega - \frac{k_i}{\omega}) + \frac{g_i k_i}{\omega}$$

and substitute into Equation (7) to give

$$[\dot{z}(\omega)] = [\phi]^{-T} [\dot{z}_i^*(\omega)] [\phi]^{-1} \quad (8)$$

The elemental mobility at forcing frequency ω is defined as

$\dot{Y}_{ij}(\omega) \equiv \partial \tilde{y}_i / \partial f_j$ and is equal to the ratio of the velocity

phasor along the coordinate i to the external force phasor along the coordinate j when no other forces are externally applied. The full mobility matrix is given by

$$[\dot{Y}(\omega)] = [\partial \tilde{y} / \partial f]_{(\omega)} = [\partial \tilde{f} / \partial \tilde{y}]_{(\omega)}^{-1} \equiv [z(\omega)]^{-1} \quad (9)$$

Therefore, using Equation (8) it is seen that

$$[\dot{Y}(\omega)] = [\phi] \left[\frac{1}{\dot{z}_i^*(\omega)} \right] [\phi]^T \equiv [\phi] [\dot{Y}_i^*(\omega)] [\phi]^T \quad (10)$$

The modal mobility of the i -th mode measured at ω is

$$\begin{aligned} \dot{Y}_i^*(\omega) &= \dot{Y}_i^{*R} + i\dot{Y}_i^{*I} = \frac{1}{\dot{z}_i^*(\omega)} = \frac{\overline{\dot{z}_i^*(\omega)}}{(\dot{z}_i^*(\omega))^2} \\ &= \frac{\dot{z}_i^{*R} - i\dot{z}_i^{*I}}{(\dot{z}_i^{*R})^2 + (\dot{z}_i^{*I})^2} = \frac{\frac{g_i k_i}{\omega} - i(\eta_i\omega - \frac{k_i}{\omega})}{(\frac{g_i k_i}{\omega})^2 + (\eta_i\omega - \frac{k_i}{\omega})^2} \end{aligned}$$

Dividing numerator and denominator of the previous equation by the generalized mass m_i

$$\dot{Y}_{i(\omega)}^* = \frac{\frac{g_i \kappa_i}{m_i \omega} - i \left(\omega - \frac{\kappa_i}{m_i \omega} \right)}{m_i \left(\frac{g_i \kappa_i}{\omega m_i} \right)^2 + m_i \left(\omega - \frac{\kappa_i}{m_i \omega} \right)^2}$$

Substituting the natural frequency of the i-th mode

$$\Omega_i = \sqrt{\frac{\kappa_i}{m_i}}$$

$$\dot{Y}_{i(\omega)}^* = \frac{\frac{g_i \Omega_i}{\omega} - i \left(\omega - \frac{\Omega_i}{\omega} \right)}{m_i \left(\frac{g_i \Omega_i}{\omega} \right)^2 + m_i \left(\omega - \frac{\Omega_i}{\omega} \right)^2}$$

Separating this equation into the real and imaginary components yields

$$\dot{Y}_{i(\omega)}^* = \frac{1}{\omega m_i} \left(\frac{\omega}{\Omega_i} \right)^2 \left[\begin{array}{c} \frac{g_i}{2} \\ \frac{g_i}{2} + \left(\frac{\omega}{\Omega_i} \right)^2 - 1 \end{array} \right] - i \left[\begin{array}{c} \left(\frac{\omega}{\Omega_i} \right)^2 - 1 \\ \frac{g_i}{2} + \left(\frac{\omega}{\Omega_i} \right)^2 - 1 \end{array} \right] \quad (11)$$

Finally, from Equation (10), the real mobility may be written as

$$[\dot{Y}_{(\omega)}^R] = [\phi] \left[\dot{Y}_{(\omega)}^{*R} \right] [\phi]^T \quad (12)$$

Reference 1 indicated that because the real modal mobilities of modes far removed from the forcing frequency become negligible compared to adjacent modes, the real mobility matrix at any frequency is ordinarily affected only by modes in close proximity to the forcing frequency. The measured real mobility matrix at a particular frequency reflects the influence of only the most dominant modes in that frequency of measurement region. Therefore, it is unrealistic to use the real mobility matrix measured at any specific frequency to determine parameters other than those associated with neighboring modes.

Reference 1 also shows that the imaginary modal mobilities of modes associated with frequencies less than the forcing frequency asymptotically approach a constant. An imaginary mobility matrix contains the effect of all lower modes in proportion to, or greater than, the magnitudes of their generalized masses. Therefore, it is impractical to use imaginary mobility matrices to evaluate properties associated with natural frequencies far above the forcing frequency.

These characteristics of the modal mobility make it impossible to determine the system parameters from the n equations in n unknowns obtained from mobility matrices measured at any two forcing frequencies.

Even if the modal mobility were amenable to determination of the system parameters, the precision of measurement which would be required to do this for most systems is impossible to achieve. The modal approach derived below avoids this problem.

DERIVATION OF THE DOMINANT MODE EIGENVALUE PROBLEM

Equation (10) may be written

$$[\dot{Y}(\omega)] = [\Phi] \left[\dot{Y}^*(\omega) \right] [\Phi]^T = \sum_{i=1}^N \dot{Y}_i^*(\omega) \{\phi\}_i \{\phi_i\}^T \quad (13)$$

where $\{\phi\}$ is a column in $[\Phi]$ and N is the order of the matrices. Define $[\Gamma] = [\Phi]^{-T}$, and Equation (8) may be written as

$$[\dot{Y}(\omega)]^{-1} = [Z(\omega)] = [\Gamma] \left[\dot{Z}^*(\omega) \right] [\Gamma]^T = \sum_{i=1}^N \frac{1}{Y_i^*(\omega)} \{\gamma\}_i \{\gamma_i\}^T \quad (14)$$

where $\{\gamma\}$ is a column in $[\Gamma]$.

Each matrix $\left[Y_i^*(\omega) \{\phi_i\} \{\phi_i\}^T \right]$ and $\left[\frac{1}{Y_i^*(\omega)} \{\gamma_i\} \{\gamma_i\}^T \right]$ in Equations (13)

and (14) is of rank one, but the summation of as many of these successive modal matrices as the order N of the matrix is a nonsingular matrix.

Similarly,

$$\begin{aligned}
 [\dot{Y}^R(\omega)] &= \sum_{i=1}^N \dot{Y}_i^{*R}(\omega) \{\phi\}_i \{\phi\}_i^T \\
 [\dot{Y}^I(\omega)] &= \sum_{i=1}^N \dot{Y}_i^{*I}(\omega) \{\phi\}_i \{\phi\}_i^T \\
 [\dot{Y}^R(\omega)]^{-1} &= \sum_{i=1}^N \frac{1}{\dot{Y}_i^{*R}(\omega)} \{\gamma\}_i \{\gamma\}_i^T \\
 [\dot{Y}^I(\omega)]^{-1} &= \sum_{i=1}^N \frac{1}{\dot{Y}_i^{*I}(\omega)} \{\gamma\}_i \{\gamma\}_i^T
 \end{aligned} \tag{15}$$

The iteration procedure used to solve the eigenvalue problem in Reference 1 employed the imaginary part of a mobility matrix measured at a frequency just above the N-th natural frequency. The method used to solve the eigenvalue problem in the present report, which was found to give more accurate results, utilizes the sum of the real parts of the mobility matrices measured near each of the natural frequencies associated with the actual model. It has been indicated previously that a measured real mobility matrix reflects the influence of only the most dominant modes in the vicinity of the forcing frequency. Therefore, summation of a discrete set of the real mobility matrices measured at forcing frequencies near the corresponding natural frequencies should contain precisely the information relevant to the model normal modes.

The eigenvalue problem may be formulated as follows. Consider the summation of the real mobility matrices measured at a discrete set of frequencies near the first NR natural frequencies. Take the inverse of this matrix and pre-multiply by a real mobility matrix measured at any frequency ω_k .

$$\begin{aligned}
& [\dot{Y}^R(\omega_k)] \left[\sum_{\omega_j = \Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} \\
&= \sum_{i=1}^{NR} \dot{Y}_i^{*R}(\omega_k) \{\phi_i\} \{\phi_i\}^T \left(\sum_{i=1}^{NR} \sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j) \{\phi_i\} \{\phi_i\}^T \right)^{-1} \\
&= [\phi] \left[\dot{Y}_i^{*R}(\omega_k) \right] [\phi]^T \left([\phi] \left[\sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j) \right] [\phi]^T \right)^{-1} \\
&= [\phi] \left[\dot{Y}_i^{*R}(\omega_k) \right] [\phi]^T [\phi]^{-T} \left[\sum_{j=1}^{NR} \frac{1}{\dot{Y}_i^{*R}(\omega_j)} \right] [\phi]^{-1} \\
&= [\phi] \left[\frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j)} \right] [\phi]^{-1}
\end{aligned} \tag{16}$$

If Equation (16) is postmultiplied by $\{\phi\}_i$, there results

$$\left[\dot{Y}^R(\omega_k) \right] \left[\sum_{\omega_j = \Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} \{\phi\}_i = [\phi] \left[\frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j)} \right] [\phi]^{-1} \{\phi\}_i$$

but $[\phi]^{-1} \{\phi\}_i$ yields a column matrix comprised of zeroes except for a 1 in the i -th position. Finally,

$$[\Phi] \left[\begin{array}{c} \dot{Y}_i^{*R}(\omega_k) \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^{-1} \{\phi\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1} \dot{Y}_i^{*R}(\omega_j)} \{\phi\}_i$$

The eigenvalue problem is finally formulated as

$$[\dot{Y}_i^R(\omega_k)] \left[\sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} \{\phi\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_i^{*R}(\omega_j)} \{\phi\}_i \quad (17)$$

If the order of multiplication is reversed in Equation (16), an eigenvalue problem is developed in which the eigenvector is the gamma vector of the i -th mode. Consider the same parameters as in Equation (16); only the order of multiplication of the matrices is changed.

$$\begin{aligned} & \left[\sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} [\dot{Y}_i^R(\omega_k)] \\ &= \left(\sum_{i=1}^{\text{NR}} \sum_{j=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_j) \{\phi_i\} \{\phi_i\}^T \right)^{-1} \left(\sum_{i=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_k) \{\phi_i\} \{\phi_i\}^T \right) \\ &= \left([\Phi] \left[\sum_{j=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_j) \right] [\Phi]^T \right)^{-1} [\Phi] \left[\dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T \\ &= [\Phi]^{-T} \left[\begin{array}{c} 1 \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^{-1} [\Phi] \left[\dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T \\ &= [\Phi]^{-T} \left[\begin{array}{c} \dot{Y}_i^{*R}(\omega_k) \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^T \end{aligned} \quad (18)$$

By definition, $[\Gamma] = [\phi]^{-T}$ and $[\Gamma]^{-1} = [\phi]^T$; substituting into Equation (18) yields

$$\begin{bmatrix} \Omega_{NR} \\ \Sigma \\ \omega_j = \Omega_1 \end{bmatrix} \dot{Y}_i^R(\omega_j) \big] [\dot{Y}_i^R(\omega_k)] = [\Gamma] \begin{bmatrix} \dot{Y}_i^{*R}(\omega_k) \\ NR \\ \Sigma \\ j=1 \end{bmatrix} [\Gamma]^{-1} \quad (19)$$

If Equation (19) is postmultiplied by $\{\gamma\}_i$, a column of $[\Gamma]$, and the same procedure is followed as was used in obtaining Equation (17), Equation (19) becomes

$$\begin{bmatrix} \Omega_{NR} \\ \Sigma \\ \omega_j = \Omega_1 \end{bmatrix} \dot{Y}_i^R(\omega_j) \big]^{-1} [\dot{Y}_i^R(\omega_k)] \{\gamma\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\begin{matrix} NR \\ \Sigma \\ \omega_j = \Omega_1 \end{matrix} \dot{Y}_i^{*R}(\omega_j)} \{\gamma\}_i \quad (20)$$

which is an eigenvalue problem with the eigenvector equal to the gamma vector of the i -th mode.

IDENTIFICATION OF STRUCTURAL PARAMETERS

A successful identification procedure, using normal mode techniques, should separate the effect of each mode in a mathematical sense, regardless of the number of stations where mobility measurements are taken on the structure. If satisfactory normal mode separation required a certain minimum number of measurement stations greater than the number of degrees of freedom chosen for the model, the most that can be expected is an approximate model, possibly including optimization procedures designed to satisfy all system constraints. This situation is considered in detail in Reference 2 in which a mathematical model is derived from test data such that identification of the structure is obtained closest to any specified analytical approximation.

Satisfactory normal mode separation requires that the values of $\dot{Z}_i^{*R}(\omega_j)$ and $\dot{Z}_i^{*I}(\omega_j)$ be independent of the number of degrees of

freedom of the model. The values of the generalized mass (m_i), the corresponding identified natural frequency (Ω_i), and the generalized stiffness as defined below are then also independent of the number of measurement stations.

$$\eta_i = \frac{\omega_k \dot{z}_i^{*I}(\omega_k) - \omega_j \dot{z}_i^{*I}(\omega_j)}{(\omega_k^2 - \omega_j^2)} \quad (21)$$

and

$$\Omega_i^2 = \omega_j \omega_k \frac{\omega_j \dot{z}_i^{*I}(\omega_k) - \omega_k \dot{z}_i^{*I}(\omega_j)}{\omega_k \dot{z}_i^{*I}(\omega_k) - \omega_j \dot{z}_i^{*I}(\omega_j)} \quad (22)$$

$$\kappa_i = \Omega_i^2 \eta_i \quad (23)$$

The two forcing frequencies (ω_k) and (ω_j) are chosen in the vicinity of the corresponding natural frequency which is available from test data. The generalized impedance of the i-th mode at forcing frequency (ω) is obtained from the generalized mobility of the i-th mode at forcing frequency (ω). It follows from Equation (13) that the modal mobilities are given by

$$\begin{aligned} \left[\dot{y}_{(\omega)}^* \right] &= [\Phi]^{-1} [\dot{Y}_{(\omega)}] [\Phi]^{-T} \\ &= [\Gamma]^T [\dot{Y}_{(\omega)}] [\Gamma] \end{aligned} \quad (24)$$

and, therefore, the orthogonality condition for gamma vectors is

$$\{\gamma\}_i^T [\dot{Y}_{(\omega)}] \{\gamma\}_i = \dot{y}_{i(\omega)}^* \delta_i^j$$

The modal impedance of the i-th mode at ω_j is

$$\dot{z}_{i(\omega_j)}^* = \frac{\dot{y}_{i(\omega_j)}^*}{|\dot{Y}_{i(\omega_j)}^*|^2} = \frac{\dot{y}_{i(\omega_j)}^{*R} - i \dot{y}_{i(\omega_j)}^*}{|\dot{Y}_{i(\omega_j)}^*|^2}$$

It follows that

$$Z_i^{*I}(\omega_j) = \frac{-Y_i^{*I}(\omega_j)}{|Y_i^{*I}(\omega_j)|^2}$$

and

$$Z_i^{*R}(\omega_j) = \frac{Y_i^{*R}(\omega_j)}{|Y_i^{*R}(\omega_j)|^2}$$

The damping coefficient for the i -th mode is most readily given by

$$g_i = \frac{\omega_j Z_i^{*R}(\omega_j)}{\kappa_i} \quad (25)$$

which follows directly from Equation (7). The damping coefficient for the i -th mode may also be obtained by

$$g_i = \left(\frac{\omega_j^2}{\Omega_i^2} - 1 \right) \frac{Z_i^{*R}(\omega_i)}{Z_i^{*I}(\omega_j)} \quad (26)$$

Using a measurement of real mobility taken precisely at resonance, the damping coefficient may be calculated using Equation (11) as

$$g_i = \frac{1}{Y_i^{*R}(\Omega_i) \Omega_i \eta_i}$$

PARAMETERS OF THE MATHEMATICAL MODEL

The elements of the influence coefficient matrix, being a measure of displacement per unit force, are independent of the number of measurement stations defining the order of the matrix. Conversely, the elements of the stiffness and mass matrices assume different values as the number of degrees of freedom of the model is changed. The identification procedure used in Reference 1 calculates both stiffness and mass matrices by summing the effects of each consecutive mode and defining the incomplete matrices as the sum up to and including a particular mode. If the order of the model

degrees of freedom is changed from ND for the structure to NR for the model, the corresponding incomplete mass and stiffness matrices will not be directly comparable, on a modal basis, to the structure mass and stiffness matrices. It is more expedient to identify the influence coefficient matrix [c] and the inverse of the mass matrix [M]. Premultiplying Equation (4) by $[\phi]^{-T}$ and postmultiplying $[\phi]^{-1}$ and taking the inverse of the resulting equation yields

$$[M] = [m]^{-1} = \sum_{i=1}^{NR} \frac{1}{m_i} \{\phi_i\} \{\phi_i\}^T \quad (28)$$

If the same operations are performed on Equation (5), the result is

$$[c] = [K]^{-1} = \sum_{i=1}^{NR} \frac{1}{\Omega^2 m_i} \{\phi_i\} \{\phi_i\}^T \quad (29)$$

Set $[c] = \frac{1}{\omega} [d]$ and using Equation (6) there results

$$\frac{1}{\omega} [gk] = \frac{1}{\omega} [\phi]^T [d] [\phi]$$

Solving for the damping matrix yields

$$[d] = \phi^{-T} [gk] \phi^{-1}$$

Substituting $[\Gamma] = [\phi]^{-T}$, $[\Gamma]^T = [\phi]^{-1}$ and $[k] = [\Omega^2 m]$ into the previous equation gives

$$[d] = [\Gamma] [g \Omega^2 m] [\Gamma]^T$$

The damping matrix can be expressed as

$$[d] = \sum_{i=1}^{NR} g_i \Omega_i^2 m_i \{\gamma_i\} \{\gamma_i\}^T \quad (30)$$

ITERATION PROCEDURE

The calculation of the modal parameters such as generalized mass, stiffness and the corresponding natural frequency requires the generalized impedance at a particular frequency for each mode under consideration. The modal impedance is a function of the generalized mobility for the same mode and forcing frequency. As indicated in Equation (24), the modal mobilities are dependent upon the matrix of gamma vectors and its transpose. The iteration process as originally formulated

in the present work involved iteration on the normal mode vectors with a subsequent inversion operation to determine the gamma vectors. This sequence introduced errors into the system, with the result that the gamma vectors did not resemble the associated gamma vectors obtained from the specimen representing the actual structure. The iterated normal mode vectors obtained from the mathematical model were extremely close, particularly at the lower modes, to the specimen, or exact, normal mode vectors. Nevertheless, any discrepancy between the model iterated modal vectors and the exact values, however small, was magnified in the inversion process, causing the gamma vectors to bear little resemblance to the specimen gamma vectors. Therefore, it was deemed advisable to iterate on the gamma vectors directly and dispense with the intermediate inversion operation.

To equalize the effect of each modal mobility in the matrix iteration Equation (20), several normalization procedures were incorporated into the method. First, each real mobility matrix was normalized on the largest element of the respective matrix. This procedure proved satisfactory except in some situations where the elements of the mobility matrices were approximately of the same magnitude but the largest elements differed in algebraic sign. This resulted in a cancellation effect among the real mobility matrices and an incorrect summation, thereby causing erroneous calculated gamma vectors. A modification to the normalization technique was applied whereby the real mobility matrices at each forcing frequency were divided by the absolute value of the largest element in the respective matrices. As a further refinement on the normalization procedure, the real mobility matrix at each forcing frequency was normalized on the root mean square associated with each respective matrix. Occasionally, these operations also caused problems in the final modal generalized mass and natural frequency calculations. For example, if a mobility matrix calculated at a particular frequency contained one element that dominated the matrix, normalization of the mobility matrix on this element would effectively submerge the influence of the matrix in the summation of the real modal mobilities. Again, the calculated modal parameters would obviously be incorrect. Similarly, if several elements of the mobility matrix measured at a specific forcing frequency were of greater magnitude than the remaining elements, the root mean square value would be affected and normalization by this value would yield a matrix wherein the elements were substantially reduced. Therefore, any such matrix would not be realistically represented in the summation of the real mobility matrices; consequently, the modal generalized mass and natural frequency would be incorrect.

Finally, each mobility matrix was multiplied by the respective forcing frequency yielding an acceleration mobility. These acceleration mobility matrices were substituted for the velocity mobilities appearing in Equation (17) and Equation (20) when iterating for the mode shapes and gamma vectors respectively. This technique was also plagued with similar difficulties that the aforementioned normalization procedures incurred. Fortunately, when the computer experiments were executed incorporating any of the previously discussed normalization methods, the conditions which yielded erroneous results were readily discernible. In these instances, the calculations for the modal generalized mass and natural frequency produced results which were obviously incorrect. For the conditions which were recognized to be in error, the computer experiments were reevaluated substituting a different normalization option. Generally, the results obtained by altering a normalization procedure yielded modal parameters which were correct.

INTERPRETATION OF ELEMENTS IN THE REDUCED MASS MATRIX

In general, it may be expected that the algebraic sum of all the elements of a reduced mass matrix from system identification will approximate the gross weight of the aircraft. Due partly to restraints, the sum of the elements should not exactly equal the gross weight, because masses at elastic restraints do not act as if they were ungrounded. Masses at pinned joints to ground do not even figure in the mass matrix because they do not move.

Individual mass elements cannot be interpreted as reflecting lumped physical weights at their assigned locations. The elements of any reduced mass matrix represent the inertial, as opposed to elastic and damping, dynamic effects of the two (for off-diagonal) degrees of freedom with which they are associated in an actual system having many more degrees of freedom than the model. The off-diagonal terms in a reduced mass matrix will usually be large and sometimes negative. The matrix will usually be fully populated.

The identified mass and stiffness matrices can be used to draw a dynamic circuit of the helicopter and, if any one were interested, it would be possible to construct an actual spring-mass system (utilizing both positive and negative springs and moments of inertia) which would be an exact physical duplication of the identified model, element by element, and would have the same natural frequencies and modal eigenvectors as the helicopter; but it would not "look" like a helicopter. Neither negative spring rates nor negative off-diagonal masses are physically unrealizable; the former are used by

Lockheed in its control system and by all light-switch manufacturers, the latter are the essential part of the dynamic antiresonant vibration isolator.

The objective is not to identify a system which "looks" like a helicopter but one which "performs" like a helicopter under various dynamic loadings. The physical interpretation of the ij -th element of the identified mass matrix, for example, is that the helicopter will generally exhibit a partial derivative of a force at i with respect to a response at j which has an effective* mass component that is the ij -th element of the identified mass matrix (similarly for the stiffness and damping matrices).

It is immaterial in the identification whether there are as many points on the structure as there are degrees of freedom in the model, or if up to three degrees of freedom (in orthogonal directions) occur at any one point. It is important only that elements in the motion vector have the properties of generalized coordinates for the holonomic model considered. An identified reduced model in which some of the displacement elements represent the orthogonal Cartesian or polar coordinates of a given structural point would look much like an identified model of a similar system with parallel coordinates of separate points.

The impedance matrix, of which the mass and stiffness matrices are terms, of a mathematical model of a larger system is a function of the size of the model, and the terms must reflect this. It was found that frequency-independent mass, stiffness and damping matrices as described can accurately reflect the responses of a continuous structure over a finite spectrum by approximating a lambda matrix the inverse of which very closely approximates the mobility. The spectral mobility matrix, even of an order that equals the number of degrees of freedom in the structure, cannot be expressed as a lambda matrix.

*Not to be confused with the formal definition of "Effective Mass" as

$$ME_{jki} \equiv \frac{\{\phi\}_i^T [m] \{\phi\}_i}{\phi_{ji} \phi_{ki}}$$

THE REDUCED MASS MATRIX

Consider the actual structure to consist of an infinite number of degrees of freedom of which R degrees of freedom are retained in the model. The mobility

$$\begin{aligned} \begin{bmatrix} [Y_{RR}] & | & [Y_{RE}] \\ \hline [Y_{ER}] & | & [Y_{EE}] \end{bmatrix} &= \begin{bmatrix} [Z_{RR}] & | & [Z_{RE}] \\ \hline [Z_{ER}] & | & [Z_{EE}] \end{bmatrix}^{-1} = \left(\begin{bmatrix} [K_{RR}] & | & [K_{RE}] \\ \hline [K_{ER}] & | & [K_{EE}] \end{bmatrix} \right. \\ &\quad \left. - \omega^2 \begin{bmatrix} [M_{RR}] & | & 0 \\ \hline 0 & | & [M_{EE}] \end{bmatrix} \right)^{-1} \end{aligned} \quad (31)$$

The model impedance is defined as the inverse of the mobility matrix in the R degrees of freedom:

$$[Z_m] \equiv [Y_{RR}]^{-1} = [Z_{RR}] - [Z_{RE}][Z_{EE}]^{-1}[Z_{ER}] = [K_m] - \omega^2 [M_m] \quad (32)$$

The stiffness of the model, $[K_m]$, is the inverse of the R x R influence coefficients:

$$[K_m] \equiv [C_{RR}]^{-1} = [K_{RR}] - [K_{RE}][K_{EE}]^{-1}[K_{ER}] \quad (33)$$

From Equation (31),

$$[Z_{RR}] = [K_{RR}] - \omega^2 [M_{RR}]$$

$$[Z_{EE}] = [K_{EE}] - \omega^2 [M_{EE}]$$

$$[Z_{RE}] = [K_{RE}]$$

$$[Z_{ER}] = [K_{ER}]$$

Substitute into Equation (32)

$$\begin{aligned}
[Z_m] &= [K_m] - \omega^2 [M_m] = [K_{RR}] - \omega^2 [M_{RR}] - [K_{RE}] \left([K_{EE}] \right. \\
&- \left. \omega^2 [M_{EE}] \right)^{-1} [K_{ER}] = [K_{RR}] - [K_{RE}] [K_{EE}]^{-1} [K_{ER}] \\
&- \omega^2 [M_{RR}] - [K_{RE}] \left([I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} [K_{EE}]^{-1} [K_{ER}] \\
&+ [K_{RE}] [K_{EE}]^{-1} [K_{ER}]
\end{aligned}$$

Substitute Equation (33). Then

$$\begin{aligned}
[M_m] &= [M_{RR}] + \frac{1}{\omega^2} [K_{RE}] \left[\left([I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} \right. \\
&- [I] \left. \right] [K_{EE}]^{-1} [K_{ER}] = [M_{RR}] + \frac{1}{\omega^2} [K_{RE}] \left[\left([I] \right. \right. \\
&- \left. \left. \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} - \left([I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right) \left([I] \right. \right. \\
&- \left. \left. \omega^2 [K_{EE}]^{-1} [M_{EE}] \right) \right]^{-1} [K_{EE}]^{-1} [K_{ER}] \\
[M_m] &= [M_{RR}] + [K_{RE}] [K_{EE}]^{-1} [M_{EE}] \left([I] \right. \\
&- \left. \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} [K_{EE}]^{-1} [K_{ER}] \tag{34}
\end{aligned}$$

Equation (34) is the "exact" R x R reduced mass matrix of a system with an infinite number of degrees of freedom. Note that $[M_m]$ is not diagonal and is a function of forcing frequency.

The frequency dependency of the "exact" reduced mass matrix simply reflects the fact that R linear differential equations with constant coefficients cannot contain enough information to exactly reflect the action of an infinite number of degrees of freedom over a spectrum containing R modes. The frequency dependency makes it impractical to use this in a linear engineering mathematical model.

The "Consistent Mass Matrix" (Reference 3), often used in finite-element dynamics work, is also based on a model stiffness matrix $[K_m]$ being the inverse of the R x R influence coefficient matrix:

$$[K_m] = [C_{RR}]^{-1} = [K_{RR}] - [K_{RE}] [K_{EE}]^{-1} [K_{ER}].$$

The kinetic energy of the structure is set equal to the

kinetic energy of the model:

$$\begin{Bmatrix} \dot{Y}_R \\ \dot{Y}_E \end{Bmatrix}^T \begin{bmatrix} [M_{RR}] & | & 0 \\ \hline 0 & | & [M_{EE}] \end{bmatrix} \begin{Bmatrix} \dot{Y}_R \\ \dot{Y}_E \end{Bmatrix} = \{\dot{Y}_R\}^T [M_{RR}] \{\dot{Y}_R\} \\ + \{\dot{Y}_E\}^T [M_{EE}] \{\dot{Y}_E\} = \{\dot{Y}_R\}^T [M_m] \{\dot{Y}_R\} \quad (35)$$

It is implicitly assumed, however, that the inertial forces occur only along the R generalized coordinates, giving

$$\begin{bmatrix} [K_{RR}] & | & [K_{RE}] \\ \hline [K_{ER}] & | & [K_{EE}] \end{bmatrix} \begin{Bmatrix} Y_R \\ Y_E \end{Bmatrix} = \begin{Bmatrix} [M_m] \{\ddot{Y}_R\} \\ 0 \end{Bmatrix}$$

which is clearly not the case but from which it follows that

$\{\dot{Y}_E\} = - [K_{EE}]^{-1} [K_{ER}] \{\dot{Y}_R\}$ in sinusoidal vibration. Sub-

stituting the above in Equation (35) gives

$$\{\dot{Y}_R\}^T [M_{RR}] + \left([K_{ER}]^T [K_{EE}]^{-T} [M_{EE}] [K_{EE}]^{-1} [K_{ER}] \right) \{\dot{Y}_R\} \\ = \{\dot{Y}_R\}^T [M_m] \{\dot{Y}_R\} \quad (36)$$

and the "Consistent Mass Matrix" is given by

$$[M_m] = [M_{RR}] + [K_{ER}]^T [K_{EE}]^{-T} [M_{EE}] [K_{EE}]^{-1} [K_{ER}] \quad (37)$$

This matrix is nondiagonal, like the "exact" reduced mass matrix, and has the advantage of being independent of frequency. However, comparison of Equation (37) with Equation (34) shows that the "Consistent Mass Matrix" reduces to the "exact" reduced mass matrix only at zero frequency; that is, in the static condition. As the frequency increases, the "Consistent Mass Matrix" yields increasingly erroneous results.

The reduced mass matrix in system identification is, like the others, nondiagonal and related to a model stiffness matrix which is the inverse of the RxR influence coefficients (as represented by the first R modes, which is accurate beyond direct measurement capability by many orders of magnitude); but the system identification reduced mass matrix also is independent of frequency and is exact at all the natural frequencies of the model, which are the first R natural frequencies of the helicopter. The system identification mass matrix is given by

$$[M_m] = [M_{RR}] + [\phi_{RR}]^{-T} [\phi_{ER}]^T [M_{EE}] [\phi_{ER}] [\phi_{RR}]^{-1} \quad (38)$$

$$\text{or } [M_m] \cong [C_{RR}]^{-1} [\phi_{RR}] \left[\frac{1}{\Omega_R^2} \right] [\phi_{RR}]^{-1} \text{ very nearly.} \quad (39)$$

At the r-th natural frequency,

$$[C_{RR}]^{-1} \left[[M_{RR}] + [K_{RE}] [K_{EE}]^{-1} \left(\frac{1}{\Omega_r^2} [I] - [C_{EE}] [M_{EE}] \right) [K_{EE}]^{-1} [K_{ER}] [C_{RR}] [M_{RR}] \right] \{\phi_{Rr}\} = \{\phi_{Rr}\} \frac{1}{\Omega_r^2} \quad (40)$$

exactly. Note in Equation (38) that the reduced system identification mass matrix is expressed in terms of the modal eigenvectors of the first R modes only but includes all the masses of the actual helicopter.

Alterations in masses on the R generalized coordinates which do not affect the modal eigenvectors are, as seen from Equation (38), exactly represented. Such alterations can substantially change natural frequencies and responses. Other types of changes which do alter the modal eigenvectors may or may not be accurately reflected in the model response depending on the degree of eigenvector effects - a limit which has not been algebraically defined for any mathematical model, whether from intuitive analysis or system identification.

That such a limit should somewhere exist is a practical engineering fact. One cannot expect to obtain the equations of a sweet pea on a rubber band, then attach it to the Golden Gate bridge and expect to find the dynamic response of the bridge (the reverse, incidentally, is equally impractical). Prudence marks the boundary between utility and uselessness.

INFORMATION LOSS IN MATRIX INVERSION

It is inevitable that there will be a loss in information in numerically obtaining the response matrix from any mathematical model, or in obtaining the mathematical model from responses, even if no deliberate error is introduced.

The following is a slight modification of a derivation by Rosanoff and Ginsburg (Reference 4). Consider the equation

$$[A]\{x\} = \{b\} \quad (41)$$

in which $[A]$ is a real symmetric nonsingular matrix. Because we calculate with numbers which have a finite number of digits, we actually solve the equation

$$([A] - [E])\{x + \delta x\} = \{b\} \quad (42)$$

where $[E]$ is an "error" matrix. Premultiplying both sides of Equation (42) by $[A]^{-1}$ and substituting $[A]^{-1}\{b\} = \{x\}$ gives

$$([I] - [A]^{-1}[E])\{x + \delta x\} = \{x\} \quad (43)$$

or

$$\{\delta x\} = \left[([I] - [A]^{-1}[E])^{-1} - [I] \right] \{x\} \quad (44)$$

Take the norm (see References 5 and 6, for example) of both sides:

$$||\{\delta x\}|| = || \left[([I] - [A]^{-1}[E])^{-1} - [I] \right] \{x\} ||$$

But the norm of the product of a matrix and a vector is less than the product of the matrix norm and the consistent vector norm:

$$||\{\delta x\}|| \leq || \left[([I] - [A]^{-1}[E])^{-1} - [I] \right] || \cdot ||\{x\}|| \quad (45)$$

or

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq || \left[([I] - [A]^{-1}[E])^{-1} - [I] \right] ||$$

Assume that $||[A]^{-1}[E]|| < 1$. From Faddeeva (Reference 5), it is well known that

$$|| ([I] - [A]^{-1}[E])^{-1} - [I] + ([A]^{-1}[E]) + ([A]^{-1}[E])^2 \dots + ([A]^{-1}[E])^k || \leq \frac{||[A]^{-1}[E]||^{k+1}}{1 - ||[A]^{-1}[E]||} \quad (46)$$

if $||[A]^{-1}[E]|| < 1$. Setting $k = 0$ gives

$$|| ([I] - [A]^{-1}[E])^{-1} - [I] || \leq \frac{||[A]^{-1}[E]||}{1 - ||[A]^{-1}[E]||} \quad (47)$$

or

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq \frac{||[A]^{-1}[E]||}{1 - ||[A]^{-1}[E]||} \leq \frac{||[A]^{-1}|| \cdot ||[E]||}{1 - ||[A]^{-1}|| \cdot ||[E]||}$$

which is identical to the result obtained by Rosanoff and Ginsburg.

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq \frac{||[A]|| \cdot ||[A]^{-1}|| \cdot ||[E]|| / ||[A]||}{1 - ||[A]|| \cdot ||[A]^{-1}|| \cdot ||[E]|| / ||[A]||} = \frac{k_n^\ell}{1 - k_n^\ell} \quad (48)$$

where, following Rosanoff et al, k_n is defined as a conditioning number and ℓ as a relative error:

$$k_n \equiv ||[A]|| \cdot ||[A]^{-1}||$$

$$\ell \equiv ||[E]|| / ||[A]||$$

Taking the number of digits in the arithmetic as $\log_{10} \frac{1}{\ell}$, the reciprocal of Equation (48) gives an estimate of the number of significant digits p .

$$p = \log_{10} ||x|| - \log_{10} ||\delta x|| \geq \log_{10} (1 - k_n^\ell) + \log_{10} \frac{1}{\ell} - \log_{10} k_n$$

but, assuming $1 \gg k_n^\ell$, this estimate may be written

$$p = \log_{10} \frac{1}{k} - \log_{10} k_n \quad (49)$$

Thus, as shown in Reference 4, the number of information digits q lost in inverting $[A]$ is approximately

$$q = \log_{10} k_n = \log_{10} ||[A]|| \cdot ||[A]^{-1}|| \quad (50)$$

This is true for any norm. However, the norm of a symmetrical positive definite matrix, subordinate to the Euclidian vector, is the maximum eigenvalue; and the maximum eigenvalue of the inverse is the reciprocal of the minimum eigenvalue of the matrix. Substituting this norm of $[A]$ into Equation (50) gives the lost digits estimate.

$$q = \log_{10} \frac{\lambda(A)_{\max}}{\lambda(A)_{\min}} \quad (51)$$

To illustrate the immense practical importance of this, consider as an example a matrix having

$$\frac{\lambda(A)_{\max}}{\lambda(A)_{\min}} = 1.72 \times 10^3$$

This is the ratio of natural frequencies in the 20x20 specimen of the helicopter used in this contract. The IBM 360 uses six hexadecimal places resulting in $16^6 - 1$ or 16777215 as the largest decimal mantissa in single precision. The inversion of the k^{-1}_m matrix with single precision on the computer results in an inverse having (estimated) $\log_{10} 16777215 - \log_{10} 1.72 \times 10^3 = 3.99$ significant digits. In other words, even starting with eight decimal places in floating point, we end up with approximately four decimal places of information in the inverse.

It is absolutely essential when dealing with test data matrices which will be inverted that the ratio of the extreme eigenvalues be minimized. Otherwise, all the physical information in the matrix is likely to be destroyed in the inversion, leaving meaningless numbers. Test data has few enough significant figures of information to begin with.

HOW TO MINIMIZE INFORMATION LOSS

A major step in this system identification process is the determination of the $\{\gamma\}$ and $\{\phi\}$ vectors by iteration. The matrix involved is the product of a mobility matrix and the inverse of another mobility matrix. This inverse presents a serious danger of information loss.

To minimize the extraneous information content of modes higher than the order of the matrix, which amounts to noise, and to narrow the spread of modal mobilities, the matrix to be inverted was made the sum of the dissipative (e.g., $[\ddot{Y}^*I]$) matrices measured near each natural frequency.

Each dissipative mobility matrix has a high information content about the dominant mode and very little information about other modes. This minimizes pollution by unwanted modes but results in a very poorly conditioned matrix. For example, the 10x10 imaginary acceleration mobility of a typical helicopter measured at 3 Hz has an extreme modal mobility ratio of 10^6 . However, the sum of mobility matrices over the frequency range is a matrix having the same modal vectors as a mobility matrix at any one frequency.

$$[\ddot{Y}_{\omega p}^*I] = \sum_{i=1}^N \ddot{Y}_{\omega p i}^*I \{\phi_i\} \{\phi_i\}^T = [\Phi] \left[\ddot{Y}_{\omega p i}^*I \right] [\Phi]^T \quad (52)$$

$$\sum_{\omega} [\ddot{Y}_{\omega}^*I] = \sum_{i=1}^n \sum_{\omega} \ddot{Y}_{\omega i}^*I \{\phi_i\} \{\phi_i\}^T = [\Phi] \left[\sum_{\omega} \ddot{Y}_{\omega}^*I \right] [\Phi] \quad (53)$$

Therefore, Equation (53) can be used as one of the matrices in the modal eigenvector equations

$$[\sum_{\omega} \ddot{Y}_{\omega}^*I]^{-1} [\ddot{Y}_{\omega}^*I] \{\gamma\}_i = \lambda \{\gamma\}_i \quad (54)$$

$$[\ddot{Y}_{\omega}^*I] [\sum_{\omega} \ddot{Y}_{\omega}^*I]^{-1} \{\phi\}_i = \alpha \{\phi\}_i \quad (55)$$

The range of values from the maximum to the minimum in $\sum_{\omega} \ddot{Y}_{\omega}^*I$ is very small compared to the range of $\ddot{Y}_{\omega i}^*I$.

If $\sum_{\omega} [Y_{\omega}^I]$ or $\sum_{\omega} [\dot{Y}_{\omega}^R]$ is used in place of $\sum_{\omega} [\ddot{Y}_{\omega}^I]$ it is necessary to normalize each of the matrices in the sum because the displacement and velocity mobilities decrease in magnitude with increased frequency. Normalization on the root mean square of the matrix elements and on the largest element absolute value were both investigated experimentally. Normalization on the RMS gave results about as satisfactory as those from acceleration mobility and is preferred over normalization on the largest element, as the latter is sensitive to errors in one term which could throw off the entire matrix. However, the differences in results, while evident, were not dramatic.

The \log_{10} of the ratio of the maximum $\sum_{\omega} \ddot{Y}_{\omega i}^{*I}$ to the minimum was generally about .75 for the 5x5 models in these experiments and generally around 1.8 for the 15 x 15 models. The 5 x 5 models performed excellently but the 15 x 15 models performed capriciously.

If the engineer could normalize so that the matrix $[\sum_{\omega} \ddot{Y}_{\omega i}^{*I}]$ is unity, information would still be lost in the inversion but certainly less information than if $\sum_{\omega} \ddot{Y}_{\omega i}^{*I}$ terms the ratio of extreme values is very high. The $\sum_{\omega} \ddot{Y}_{\omega i}^{*I}$ terms

are not the eigenvalues of $\sum_{\omega} [\ddot{Y}_{\omega}^I]$. The only matrix which has a unit eigenvalue matrix is the unit matrix itself; it follows therefore that some information is always lost in the numerical inversion of any matrix other than unity.

The matrix we wish to invert is

$$\sum_{\omega} [Y_{\omega}^I] = [\Phi] \left[\sum_{\omega} \ddot{Y}_{\omega i}^{*I} \right] [\Phi]^T \quad (56)$$

Express the modal vector matrix in terms of its own eigenvectors $[J]$ and its own eigenvalues λ_{ϕ} (that is, $[J]$ is the eigenvector matrix of the eigenvector matrix of $[[k]^{-1}[m]]$).

$$[\Phi] = [J] [\lambda_{\phi}] [J]^{-1} \quad (57)$$

Substitute Equation (57) into Equation (56).

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I] = [J] [\lambda_{\phi}] [J]^{-1} [\Sigma_{\omega i}^I] [J]^{-T} [\lambda_{\phi}] [J]^T \quad (58)$$

invert,

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I]^{-1} = [J]^{-T} \left[\frac{1}{\lambda_{\phi}} \right] [J]^T \left[\frac{1}{\Sigma_{\omega i}^I} \right] [J] \left[\frac{1}{\lambda_{\phi}} \right] [J]^{-1} \quad (59)$$

The only operation on the eigenvectors [J] between Equation (58) and Equation (59) was to change relative positions; all the inversions were of diagonal matrices. As a diagonal matrix is a matrix of its own eigenvalues, having the unit matrix for eigenvectors, the central term may be treated rigorously as an eigenvalue matrix. The matrix of Equation (58) may be substituted for [A] in Equation (41), and in Equation (50), we can consider [A] as the product of the three matrices of Equation (56).

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I] = [A] = [\Phi] [\Sigma_{\omega i}^{*I}] [\Phi]^T \quad (60)$$

It is well known that

$$||[A]|| \leq ||[\Phi]|| \cdot ||[\Sigma_{\omega i}^{*I}] [\Phi]^T|| \quad (61)$$

and that

$$||[\Sigma_{\omega i}^{*I}] [\Phi]^T|| \leq ||[\Sigma_{\omega i}^{*I}]|| \cdot ||[\Phi]^T|| \quad (62)$$

Therefore

$$||[A]|| \leq ||[\Phi]|| \cdot ||[\Sigma_{\omega i}^{*I}]|| \cdot ||[\Phi]^T|| \quad (63)$$

At this point we wish to substitute eigenvalues, but $[\Phi]$ is not symmetric so $||[\Phi]|| \neq |\max \lambda_{\phi}|$. Rather, $||[\Phi]|| \geq |\max \lambda_{\phi}|$. Consider, therefore, the eigenvalues λ_b of $[\Phi]^T [\Phi]$ which is symmetrical.

$$[\Phi]^T [\Phi] = [L] [\lambda_b] [L]^{-1} = [L] [\lambda_b] [L]^T \quad (64)$$

where [L] is the orthogonal matrix of eigenvectors of $[\Phi]^T [\Phi]$.

$$||[\Phi]|| = |\max \sqrt{\lambda_b}| \quad (65)$$

Substitute Equation (65) into Equation (63).

$$||[A]|| \leq |\max \lambda_b| \cdot |\max \Sigma \ddot{Y}_{\omega i}^{*I}| \quad (66)$$

Using Equation (51), the number of digits lost in inverting $\Sigma [Y_{\omega}^*]$ is approximated by

$$\begin{aligned} q &\approx \log_{10} \frac{|\max \lambda_b| \cdot |\max \Sigma \ddot{Y}_{\omega i}^{*I}|}{|\min \lambda_b| \cdot |\min \Sigma \ddot{Y}_{\omega i}^{*I}|} \\ &= \log_{10} \frac{|\max \lambda_b|}{|\min \lambda_b|} + \log_{10} \frac{|\max \Sigma \ddot{Y}_{\omega i}^{*I}|}{|\min \Sigma \ddot{Y}_{\omega i}^{*I}|} \end{aligned} \quad (67)$$

$[\lambda_b]$ would equal a scalar times the unit matrix only if the modal vectors $\{\phi\}$ were orthogonal (i.e., $\{\phi_i^T\} \{\phi_j\} = 0$), a condition which could occur only in the academic cases of uniform mass: $[m] = [m][I]$. In this case, the loss of information digits would be indicated by

$$q \approx \log_{10} \frac{|\max \ddot{Y}_{\omega i}^{*I}|}{|\min \ddot{Y}_{\omega i}^{*I}|} \quad (68)$$

and only in this case could zero information loss be achieved by normalizing the matrices such that $|\max \ddot{Y}_{\omega i}^{*I}| / |\min \ddot{Y}_{\omega i}^{*I}| = 1$.

But the case is trivial, for if it were true, an inversion would be unnecessary as ϕ would be the eigenvector matrix of $\Sigma [Y_{\omega}^*]$.

If the mass distribution is not uniform diagonal but the engineer could so normalize the matrices in the summation so that $|\max \ddot{Y}_{\omega i}^{*I}| / |\min \ddot{Y}_{\omega i}^{*I}| = 1$, it is seen from Equation (67) that there would still be a loss of information digits approximated by

$$q \approx \log_{10} \frac{|\max \lambda_b|}{|\min \lambda_b|} \quad (69)$$

The ratio $|\max \lambda_b|/|\min \lambda_b|$ increases with the order of the mobility matrix; that is, with the number of degrees of freedom of the model. It follows, therefore, that there is an upper limit to the size of a physically meaningful reduced complete model regardless of normalization of the matrices in the summation.

As a crude "rule of thumb", Figure 1 shows the trend in the reliability of the inversion of $\Sigma_{\omega} \ddot{Y}_{\omega}^I$.

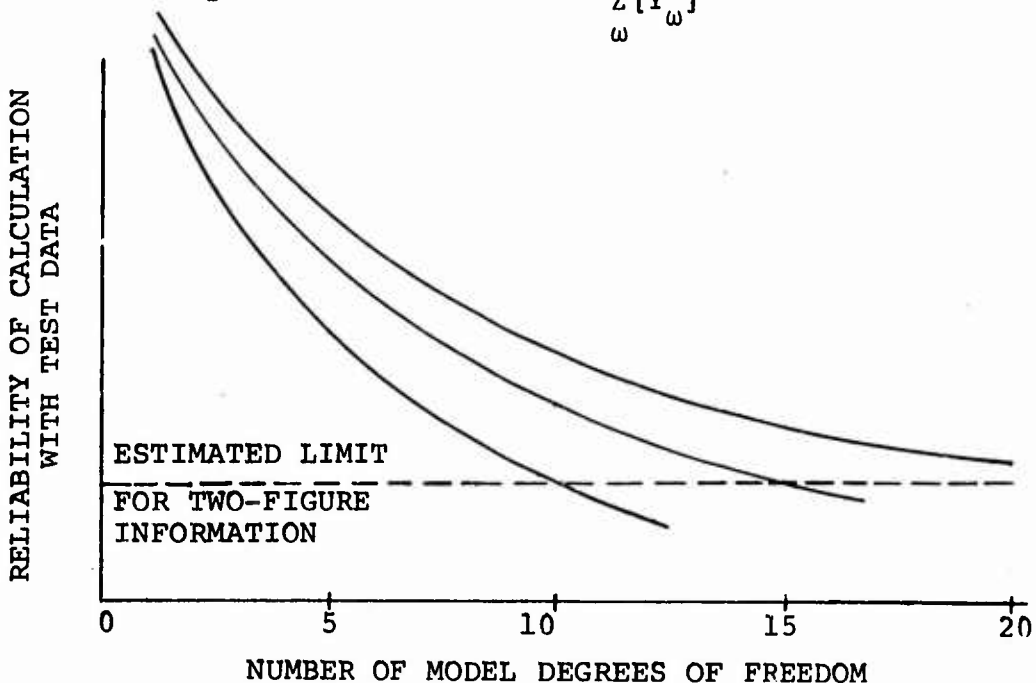


Figure 1. Reliability of the Inversion of $\Sigma_{\omega} \ddot{Y}_{\omega}^I$.

It is seen that the reliability of the calculation becomes questionable above 10 or 15 degrees of freedom. This does not mean that accurate identifications cannot be made using the iterative step for modes of, say, 20 degrees of freedom but, rather, that any one calculation has a higher probability of failure.

In passing, it should be noted that the treatment of bounds using matrix norms, as above, opens up some highly promising avenues of research on the reliability of many helicopter

theoretical calculations as well as on the reliability of the processing of test data in general. Whether, for example, some of the conventional methods of processing strain gage data yield physically meaningful results is open to question in the light of the above method.

WHEN A CALCULATION FAILS

The most common mode of failure of iteration on $[\ddot{Y}_\omega^I] (\Sigma [\ddot{Y}_\omega^I])^{-1}$

is catastrophic, producing such absurd values for one or more generalized masses as negative numbers or unusually large numbers. This is signified also by a very large number of iterations required for convergence on one or more modes. Failures almost never occur with small number of degrees of freedom (e.g., five), and an identification which is quite accurate with one seed may, in the larger models, diverge with another seed.

This phenomenon results from the fact that the significant effect of error is not insidious accumulation of inaccuracies in the generalized masses but, rather, information destruction in the inversion. Fortunately, it is usually very obvious to the engineer when an identification fails on the computer, and corrective measures may be taken without rerunning the test on the helicopter.

A most obvious and effective corrective measure is to eliminate one or more of the degrees of freedom. This can be done on the computer, as the program is written so that the system may be instructed to select any of the available data which is in digital form on tape. The size of the model and the number of modes covered are consequently reduced. It is possible also to eliminate any mode, not just the highest, if it appears that a certain mode contributes little information - a local resonance, for example, in which only a small portion of the helicopter is significantly responding.

The computer experiments included a local resonance in the form of a mode in which only the most forward station showed substantial motion. When this station was not included in the identification, but the local resonance associated with its movement was included, then, as expected, there were evidences of failure in generalized mass calculation. The computer was attempting to identify a natural mode for which the input mobility data showed a largely nonresponding helicopter. This situation would be detected from the mobility plots before committing the data, as it is very apparent in the dissipative mobility spectra. The

ability to handle local resonances, or dispose of them when required, is important to a practical identification because all real structures have them. In fact, as the number of degrees of freedom of a simulated structure are realistically increased, the modal density usually increases more rapidly than the simple mathematics of uniform chains would lead one to believe. When that degree of freedom which is the predominant motion of a local resonance is eliminated, the mode it causes should be eliminated also; the mobility spectra plots for the included degrees of freedom would indicate this by an insignificant peak.

THE Γ MATRIX AND MODAL PARAMETERS

The dominant modal vector at frequency ω_i , near the i -th natural frequency Ω_i , is given by Equation (55) and the i -th gamma vector by iteration on the transpose, Equation (54). The modal mobility is obtained from

$$\{\gamma_i\}_{(ITR)}^T [Y_{\omega_i}^I] \{\gamma_i\}_{(ITR)} = \ddot{Y}_{i\omega_i}^{*I} \quad (70)$$

where $\{\gamma_i\}_{(ITR)}$ is the vector from iteration (Equation 54).

It is impractical to attempt the calculation using $\{\gamma_i\}$ from $[\Phi]^{-T}$ because of information loss in the inversion, as shown in Equation (59). The dominant mode is the only one used, of course, as there is negligible information content in $[Y_{\omega_i}^I]$ about modes other than the i -th. Therefore,

$$[Y_{\omega_i}^I] \approx \ddot{Y}_{i\omega_i}^{*I} \{\phi_i\} \{\phi_i\}^T \approx \ddot{Y}_{i\omega_i}^{*I} \{\phi_i\}_{(ITR)} \{\phi_i\}_{(ITR)}^T \quad (71)$$

and $\{\gamma_i\}_{(ITR)}^T \{\phi_i\}_{(ITR)} = 1$ is forced.

A peculiar situation often occurred when a calculation diverged: it was noticed that the natural frequency of the "bad" mode was usually identified with great accuracy although the calculated generalized mass was absurd, often negative, and negative calculated values of $\ddot{Y}_{i\omega_i}^{*R}$ often occurred.

The key here is the occurrence of negative values of $\dot{Y}_{i\omega_i}^{*R}$. Ideally, $[\dot{Y}^R]$ is a positive definite matrix and cannot, theoretically, be negative definite on grounds that it represents the dissipation, not a source, of energy. For any positive definite matrix B, $\{x\}^T[B]\{x\}$ is a positive number regardless of the choice of the vector $\{x\}$. The fault for negative values of $\dot{Y}_{i\omega_i}^{*R}$, which are physically impossible,

cannot therefore be laid solely to $\{\gamma\}$, and therefore to the loss of information in the inverse of $\sum_{\omega} [\dot{Y}_{\omega}^I]$, because

$\{\gamma_i\}^T[\dot{Y}^R]\{\gamma_i\}$ must be positive even for arbitrary $\{\gamma_i\}$ if $[\dot{Y}^R]$ is, as it is supposed to be, positive definite. We are forced to conclude that numerical errors can act in such a way as to make $[\dot{Y}^R]$ not positive definite.

The mobility $[\dot{Y}_{\omega_i}^{*R}]$ is very nearly equal to the positive semi-definite matrix $\left[\dot{Y}_{i\omega_i}^{*R} \{\phi_i\} \{\phi_i\}^T \right]$ in which $\dot{Y}_{i\omega_i}^{*R}$ is necessarily positive. Then

$$\{\gamma_i\}^T [\dot{Y}_{i\omega_i}^{*R} \{\phi_i\} \{\phi_i\}^T] \{\gamma_i\} = \dot{Y}_{i\omega_i}^{*R} (\{\gamma_i\}^T \{\phi_i\})^2 \quad (72)$$

But $\{\gamma_i\}$ and $\{\phi_i\}$ are composed of real numbers, as opposed to imaginary or complex numbers, which makes $\{\gamma_i\}^T \{\phi_i\}$ real and $(\{\gamma_i\}^T \{\phi_i\})^2$ real and positive even for arbitrary elements in $\{\gamma\}$. The dominance of $[\dot{Y}_{\omega_i}^{*R}]$ by one mode is therefore not a cause of calculating negative values of $\dot{Y}_{i\omega_i}^{*R}$.

The calculation of absurd values of $\dot{Y}_{i\omega}^{*R}$ is nevertheless due mainly to information loss in inverting $\sum_{\omega} [\dot{Y}_{\omega}^I]$ (or other normalized mobility matrices having similar properties), which results in poor eigenvectors in the iteration. Examination of the computer experiments shows that errors in the $[Y^R]$ or $[Y^I]$ matrices are not sufficient to cause as erratic results as have sometimes been observed if the $\{\gamma\}$ vectors in $\{\gamma\}^T [Y] \{\gamma\}$ are accurate. In the "bad" cases, the $\{\gamma\}$ vectors from iteration are invariably very bad. The reason for the occasional negative calculated values of $\dot{Y}_{i\omega}^{*R}$ is, in part,

that errors in [Y] can cause the matrix to not be positive definite. For example, in Computer Experiment 188 a nine-point identification with error yielded good results but the same identification with a different seed (Computer Experiment 184) gave poor results which included negative $\dot{Y}_{i\omega}^{*R}$ for

the seventh mode. The errors by chance happened to act in such a way in Experiment 184 that excessive information was lost in the inverse, as indicated by iterations that failed to converge. The principal minor associated with the eighth and ninth positions in mobilities dominated by the seventh mode was found to be negative in the bad case (Experiment 184), due to a peculiar accumulation of random errors, which, of course, meant that the mobility was no longer positive definite, as in pure theory, and could give negative values of $\{\gamma\}_i^T [Y_{\omega_i}] \{\gamma_i\}$. However, precise $\{\gamma\}$ vectors would not have

caused the negative values of $\dot{Y}_{i\omega}^{*R}$ even with [Y] not being positive semidefinite.

Calculation of physically meaningless values of $Y_{i\omega}^{*R}$, and therefore of M^* , is caused primarily by information loss in inversion.

The reason for fairly accurate identifications of natural frequencies even when the generalized mass identifications are poor lies in the fact that ω_j and ω_k in Equation (22) are taken near Ω_i ; therefore,

$$\frac{\omega_j \dot{Z}_{i\omega_k}^{*I} - \omega_k \dot{Z}_{i\omega_j}^{*I}}{\omega_k \dot{Z}_{i\omega_k}^{*I} - \omega_j \dot{Z}_{i\omega_j}^{*I}} \approx 1 \quad (73)$$

and

$$\Omega_i^2 = \omega_j \omega_k \frac{\omega_j \dot{Z}_{i\omega_k}^{*I} - \omega_k \dot{Z}_{i\omega_j}^{*I}}{\omega_k \dot{Z}_{i\omega_k}^{*I} - \omega_j \dot{Z}_{i\omega_j}^{*I}} \approx \omega_j \omega_k = (\Omega_i - \delta\omega_j)(\Omega_i + \delta\omega_k)$$

$$\omega_j \omega_k = \Omega_i^2 + \Omega_i(\delta\omega_k - \delta\omega_j) - \delta\omega_j \delta\omega_k$$

But $\delta\omega_k \ll \delta\omega_j$ so $\Omega_i^2 \approx \omega_j \omega_k$.

IDENTIFIED GENERALIZED MASSES

Typical generalized mass identifications are shown in Tables I through VI. Note in Tables I, III and V that the generalized mass of the first mode identified for a reduced model with no experimental error has always been less than the first mode generalized mass calculated from the modal vector and mass matrix of the specimen. This is not true of other modes.

Tables I and II show results of two different five-point models. No outstanding differences between the models is evident. Model 9A produced acceptable results, as shown in Table III, for different distribution of random error but Model 9B, as shown in Table IV, worked with some seeds and failed with other seeds. The failed experiments of Table IV, Computer Experiments 168 and 184, yielded drastically unrealistic values of generalized mass for most of the modes.

Table V shows a twelve-point model identification which failed only in the eighth mode. Computer Experiment 178 is identical to Computer Experiment 169 except that in the former, the computer was instructed to skip the eighth mode and, instead, operate on tape data for the thirteenth mode which resulted in satisfactory identification.

Using different stations for a twelve-point model, as shown in Table VI, produced proper identification of all modes, including the eighth, with various error distributions.

Information loss in the inversion of mobility matrices is the primary cause of such failures, as shown in Computer Experiments 168, 184 and 169. The averaging of mobility test data, properly done, would greatly minimize the chances of such identification failures. Test data averaging is the customary practice. These computer experiments did not take advantage of averaging experiments.

TABLE I. IDENTIFICATION OF GENERALIZED MASSES,
5 X 5 MODEL* OF 20 X 20 SPECIMEN

| | | | | | | | |
|----------------------------|------|---|--------|--------|--------|--------|--------|
| Computer Experiment Number | | 152 | 151 | 157 | 160 | 182 | 1** |
| Random Amp Error | | 0 | +5% | +5% | +5% | +5% | 0 |
| Bias Amp Error | | 0 | +5% | +5% | +5% | +5% | 0 |
| Random Phase Error | | 0 | +1 | +1° | +1° | +1° | 0 |
| Seed | | - | 246 | 221 | 195 | 327 | - |
| Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | | | | |
| 0 | 1 | 7.9910 | 7.3594 | 7.3834 | 7.8421 | 8.2330 | 8.5341 |
| 120 | 2 | 4.6248 | 3.8247 | 4.5951 | 4.1440 | 4.0594 | 4.4491 |
| 220 | 3 | .4951 | .4618 | .4771 | .4653 | .4729 | .4951 |
| 340 | 4 | 1.0897 | 1.0372 | 1.0657 | 1.0366 | 1.0440 | 1.0872 |
| 460 | 5 | .6463 | .5869 | .6247 | .6691 | .6131 | .6302 |
| * Model 5A | | | | | | | |
| ** From 20 x 20 Model | | | | | | | |

TABLE II. IDENTIFICATION OF GENERALIZED MASSES,
5 X 5 MODEL* OF 20 X 20 SPECIMEN

| | | | | | | |
|----------------------------|-------------------|------------|--|--------|--------|--------|
| Computer Experiment Number | 159 | 170 | 183 | 1** | | |
| Random Amp Error | <u>+5%</u> | <u>+5%</u> | <u>+5%</u> | 0 | | |
| Bias Amp Error | +5% | +5% | +5% | 0 | | |
| Random Phase Error | <u>+1°</u> | <u>+1°</u> | <u>+1°</u> | - | | |
| Seed | 221 | 246 | 128 | - | | |
| | Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | | |
| | 0 | 1 | 7.4385 | 7.3210 | 7.8179 | 8.5341 |
| | 100 | 2 | 4.4545 | 4.1824 | 4.3797 | 4.4491 |
| | 200 | 3 | .4724 | .4620 | .4596 | .4951 |
| | 320 | 4 | 1.0769 | 1.0277 | 1.0233 | 1.0872 |
| | 460 | 5 | .6912 | .5945 | .6360 | .6302 |
| * Model 5B | | | | | | |
| ** From 20 x 20 Model | | | | | | |

TABLE III. IDENTIFICATION OF GENERALIZED MASSES,
9 X 9 MODEL* OF 20 X 20 SPECIMEN

| | | | | | | | |
|----------------------------|------|---|--------|--------|--------|--------|--------|
| Computer Experiment Number | 180 | 156 | 162 | 179 | 187 | 1** | |
| Random Amp Error | 0 | +5% | +5% | +5% | +5% | 0 | |
| Bias Amp Error | 0 | +5% | +5% | +5% | +5% | 0 | |
| Random Phase Error | 0 | +1° | +1° | +1° | +1° | 0 | |
| Seed | - | 287 | 50 | 315 | 492 | - | |
| Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | | | | |
| 0 | 1 | 7.9538 | 7.3776 | 7.5946 | 8.2378 | 7.4531 | 8.5342 |
| 30 | 2 | 4.5889 | 4.1130 | 4.2450 | 4.6233 | 4.2020 | 4.4491 |
| 100 | 3 | .4938 | .4671 | .4821 | .4614 | .4656 | .4951 |
| 160 | 4 | 1.0863 | 1.0507 | 1.0368 | 1.0129 | 1.0785 | 1.0872 |
| 220 | 5 | .6350 | .6164 | .6044 | .6102 | .5971 | .6302 |
| 280 | 6 | .7457 | .7049 | .6983 | .7227 | .7239 | .7429 |
| 340 | 7 | 1.1746 | 1.1204 | 1.1332 | 1.1064 | 1.0968 | 1.1769 |
| 400 | 8 | 1.5002 | 1.3770 | 1.4070 | 1.4193 | 1.4783 | 1.4683 |
| 460 | 9 | .6593 | .6576 | .5507 | .6235 | .5737 | .7866 |
| * Model 9A | | | | | | | |
| ** From 20 x 20 Model | | | | | | | |

TABLE IV. IDENTIFICATION OF GENERALIZED MASSES,
9 X 9 MODEL* OF 20 X 20 SPECIMEN

| | | | | | | | |
|----------------------------|----------------|------|---|--------|---------|---------|--------|
| Computer Experiment Number | | 161 | 188 | 168 | 184 | 1** | |
| Random Amp Error | | ±5% | ±5% | ±5% | ±5% | 0 | |
| Bias Amp Error | | +5% | -5% | +5% | +5% | 0 | |
| Random Phase Error | | ±1 | ±1 | ±1 | ±1 | - | |
| Seed | | 287 | 206 | 395 | 619 | - | |
| | Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | | | |
| | 0 | 1 | 7.4445 | 7.5891 | 7.6797 | 7.0969 | 8.5341 |
| | 60 | 2 | 4.2851 | 4.4084 | 23.2234 | 4.5730 | 4.5615 |
| | 120 | 3 | .4741 | .4545 | .6876 | .4314 | .4951 |
| | 180 | 4 | 1.0194 | 1.0226 | 28.5896 | 1.0968 | 1.0872 |
| | 240 | 5 | .6343 | .6740 | .5667 | -7.9847 | .6302 |
| | 280 | 6 | .7020 | .6987 | -8.5143 | .5237 | .7429 |
| | 320 | 7 | 1.1877 | 1.0711 | -.0080 | .0125 | 1.1769 |
| | 400 | 8 | 1.2510 | 1.7815 | .1256 | -.2199 | 1.4683 |
| | 460 | 9 | .9347 | .9398 | -.0159 | -.0810 | .9836 |
| * Model 9B | | | | | | | |
| ** From 20 x 20 Model | | | | | | | |

TABLE V. IDENTIFICATION OF GENERALIZED MASSES,
12 X 12 MODEL* OF 20 X 20 SPECIMEN

| Computer Experiment Number | | 169 | 178 | 1** |
|----------------------------|------|---|--------|--------|
| Random Amp Error | | +5% | +5% | 0 |
| Bias Amp Error | | +5% | +5% | 0 |
| Random Phase Error | | +1° | +1° | 0 |
| Seed | | 492 | 492 | - |
| Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | |
| 0 | 1 | 7.4551 | 7.4629 | 8.5341 |
| 60 | 2 | 4.1298 | 3.9789 | 4.4491 |
| 100 | 3 | .4587 | .4657 | .4951 |
| 120 | 4 | 1.0446 | 1.0376 | 1.0872 |
| 160 | 5 | .5950 | .5802 | .6302 |
| 200 | 6 | .6869 | .6975 | .7429 |
| 240 | 7 | 1.2036 | 1.2044 | 1.2569 |
| 280 | 8 | -7.9616 | | 2.0521 |
| 320 | 9 | .9410 | .9118 | .9836 |
| 370 | 10 | .0425 | .0428 | .0432 |
| 430 | 11 | .1718 | .1752 | .1723 |
| 460 | 12 | 1.0012 | 1.0037 | 1.0480 |
| | 13 | - | .7924 | .5724 |
| * Model 12D | | | | |
| ** From 20 x 20 Model | | | | |

TABLE VI. IDENTIFICATION OF GENERALIZED MASSES,
12 X 12 MODEL* OF 20 X 20 SPECIMEN

| | | | | | | |
|----------------------------|------|---|--------|--------|--------|--------|
| Computer Experiment Number | | 150 | 149 | 155 | 163 | 1** |
| Random Amp Error | | 0 | +5% | +5% | +5% | 0 |
| Bias Amp Error | | 0 | +5% | +5% | +5% | 0 |
| Random Phase Error | | 0 | +1° | +1° | +1° | 0 |
| Seed | | - | 23 | 492 | 87 | - |
| Stations (In.) | Mode | Generalized Masses (Lb-Sec ² /In.) | | | | |
| 0 | 1 | 7.9718 | 7.7160 | 7.2917 | 7.4071 | 8.5342 |
| 30 | 2 | 4.6071 | 4.5010 | 4.2722 | 4.3406 | 4.4491 |
| 60 | 3 | .4941 | .4640 | .4682 | .4611 | .4951 |
| 100 | 4 | 1.0857 | 1.0499 | 1.0625 | 1.0425 | 1.0872 |
| 140 | 5 | .6348 | .6094 | .5958 | .5936 | .6302 |
| 180 | 6 | .7441 | .7155 | .6930 | .7097 | .7429 |
| 220 | 7 | 1.1765 | 1.1433 | 1.1101 | 1.1278 | 1.1769 |
| 260 | 8 | 1.4158 | 1.3467 | 1.3225 | 1.3454 | 1.4115 |
| 300 | 9 | .7808 | .7329 | .7395 | .7362 | .7866 |
| 340 | 10 | .0430 | .0419 | .0422 | .0422 | .0432 |
| 400 | 11 | .1705 | .1596 | .1665 | .1689 | .1723 |
| 460 | 12 | .9112 | .5712 | .8417 | 1.0946 | 1.3235 |
| * Model 12B | | | | | | |
| ** From 20 x 20 Model | | | | | | |

RESPONSE FROM IDENTIFIED MODEL

Figures 2 through 7 portray typical acceleration response obtained from the various models investigated in the present study. In each instance, the exact curve was obtained from the twenty-point structure with zero error. Figure 2 indicates the effect of random number seed for a typical five-point model. Figure 3 presents the results obtained for one of the nine-point models considered in the investigation. Figure 4 portrays the effect of random number seed on the twelve-point model. All the computer experiments which considered error used a +5% random, 5% bias and a 1° phase error.

Figure 5 presents the effect of model variation on the acceleration response. The models varied in that different spanwise masses were considered. Model 5A utilized stations 0, 120, 220, 340 and 460 (inches) whereas model 5B consisted of stations 0, 100, 200, 320, and 460 (inches). Figure 6 presents the effect of model for the nine-point model. The model 9A consisted of stations 0, 30, 100, 160, 220, 280, 340, 400 and 460 (inches). Model 9B included stations 0, 60, 120, 180, 240, 280, 320, 400 and 460 (inches). The twelve-point model 12B used stations 0, 30, 60, 100, 140, 180, 220, 260, 300, 340, 400 and 460 (inches) whereas model 12E utilized stations 0, 30, 60, 100, 120, 160, 200, 260, 280, 340, 400, 460 (inches). For each model the computer experiments were executed using the same random number seed and the aforementioned errors were incorporated.

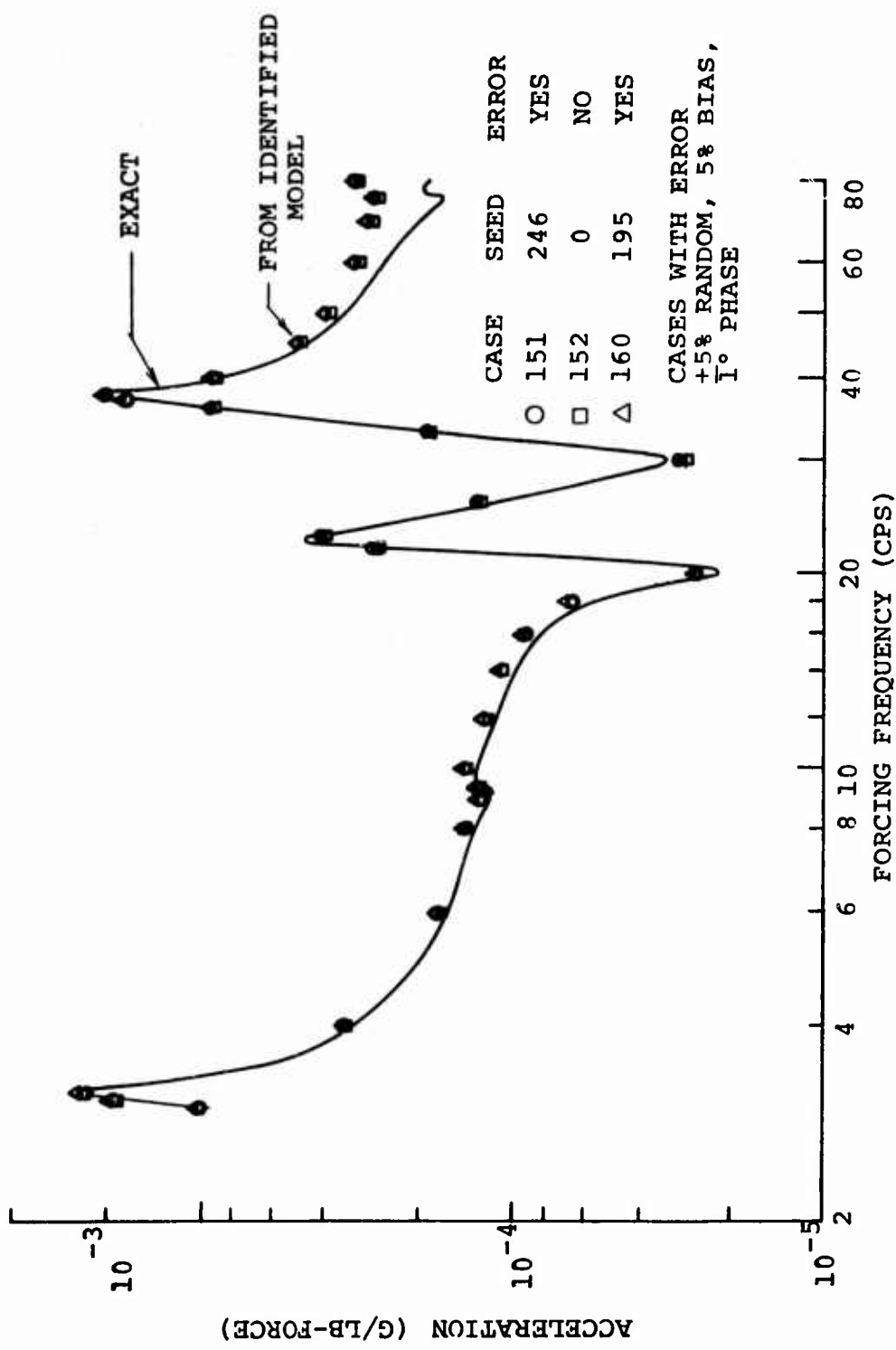


Figure 2. Five-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at Hub.

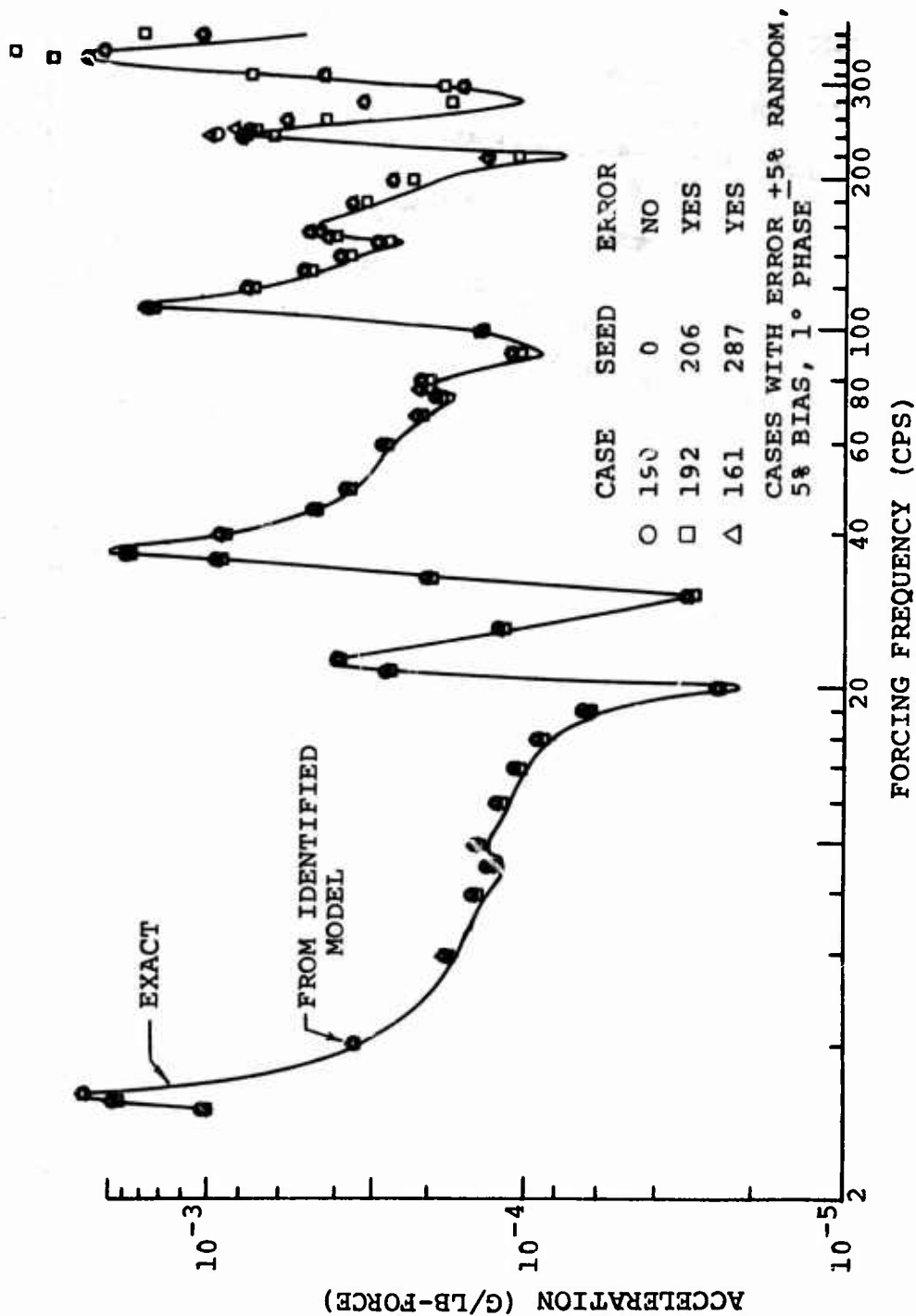


Figure 3. Nine-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at Hv.D.

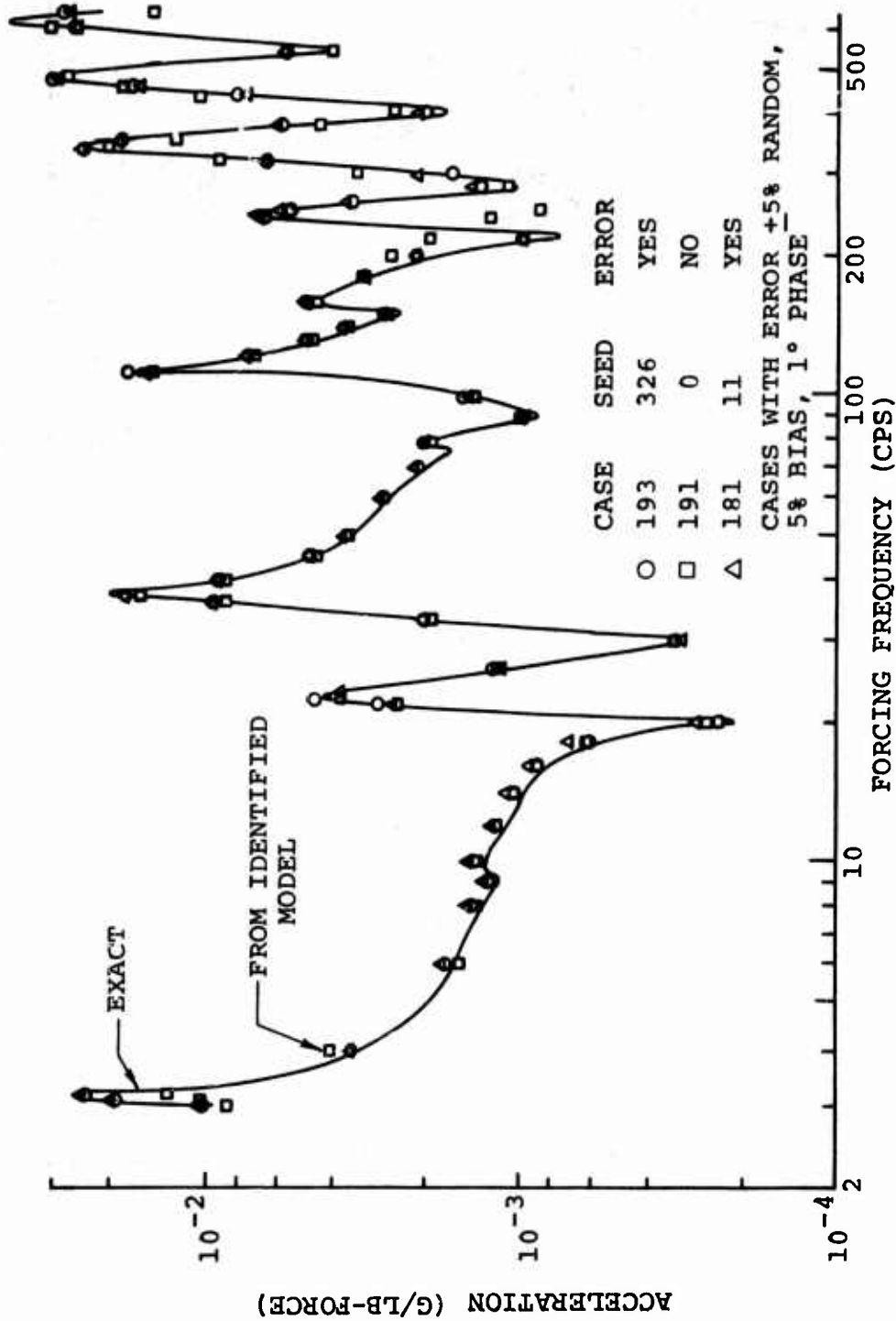


Figure 4. Twelve-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at Hub.

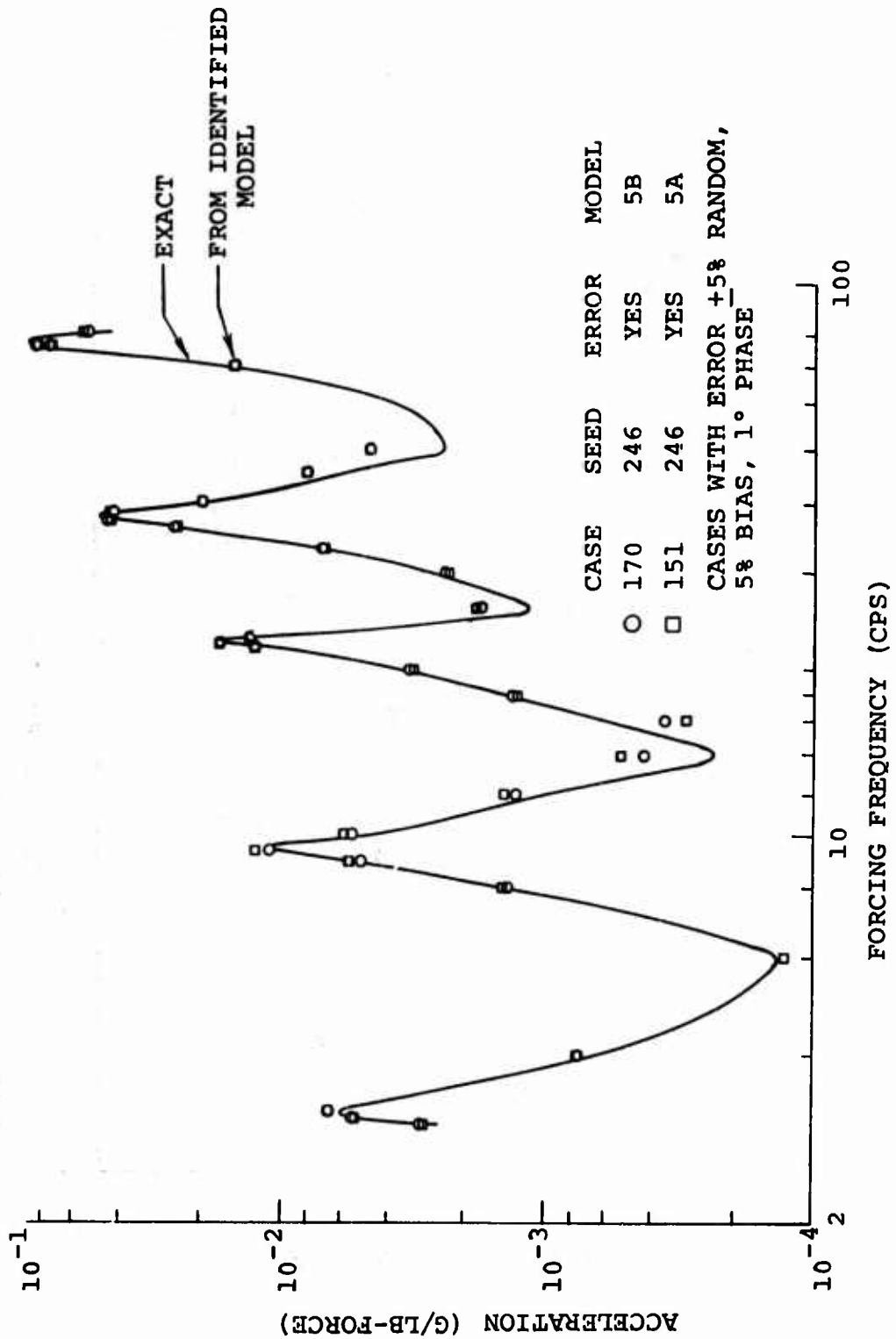


Figure 5. Five-Point Model Response, Effect of Model; Driving Point at Station 1.

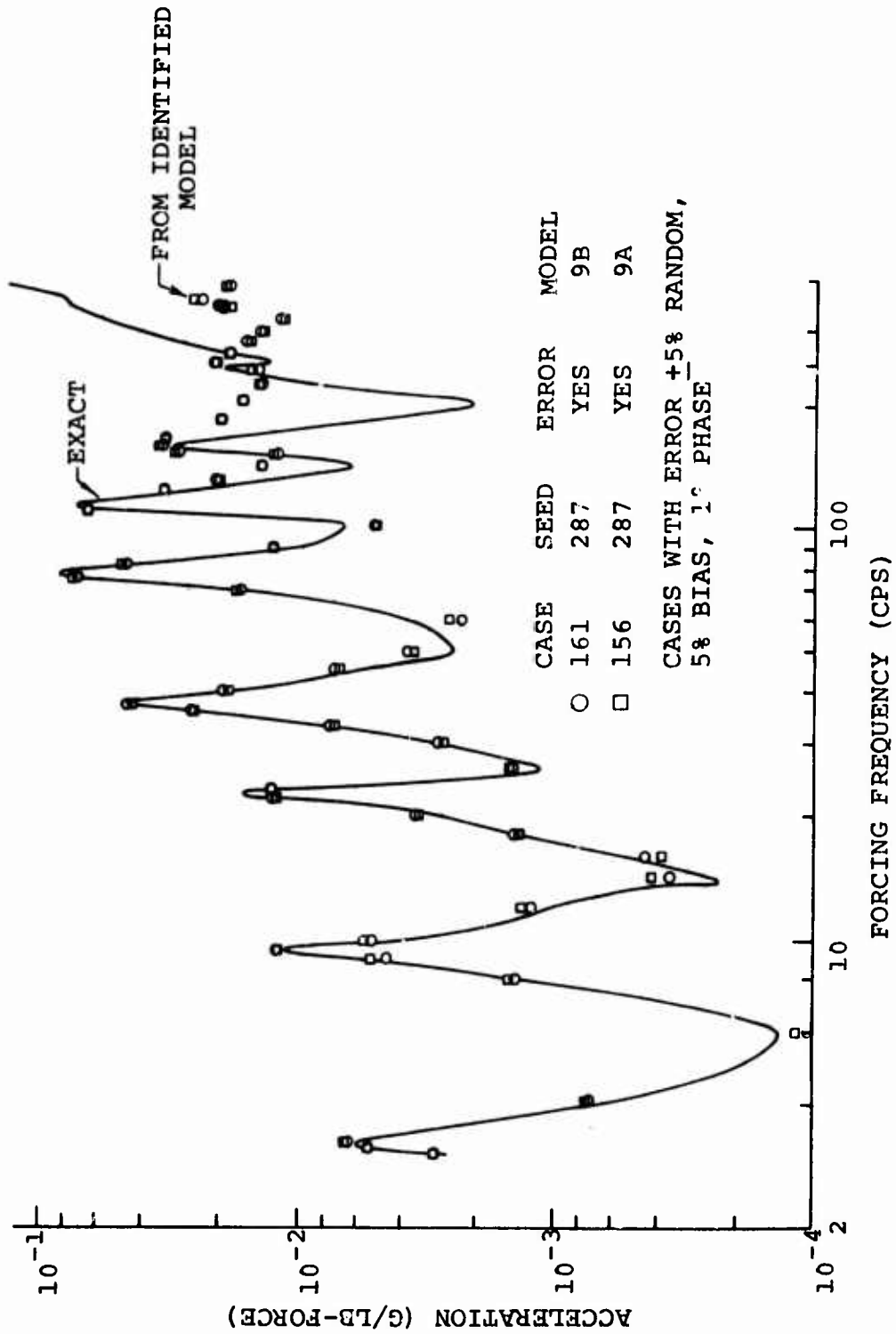


Figure 6. Nine-Point Model Response, Effect of Model; Driving Point at Station 1.

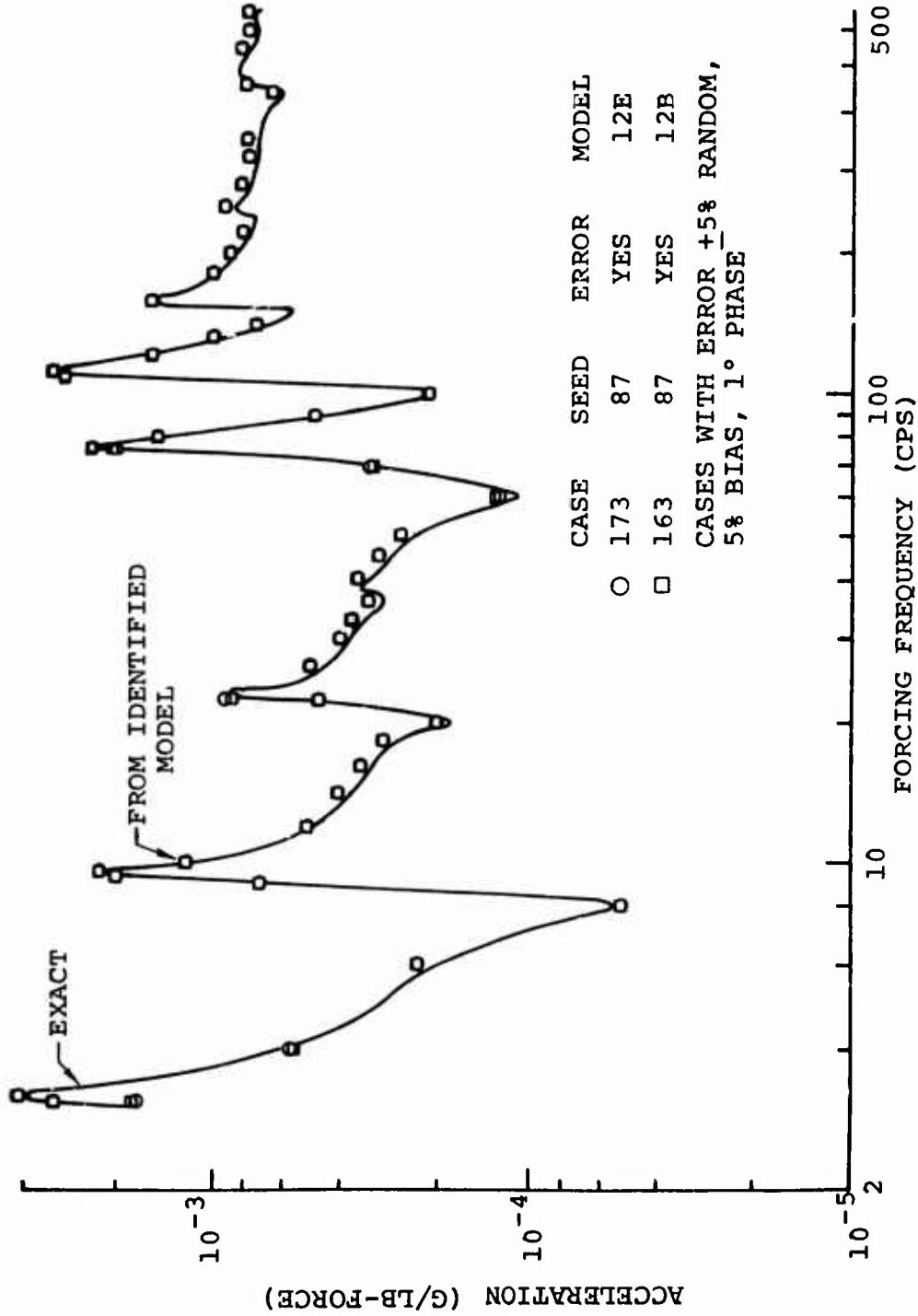


Figure 7. Twelve-Point Model Response, Effect of Model; Driving Point at Station 3.

CONCLUSIONS

1. The equations of motion for a structure may be obtained using only impedance-type test data without the use of an intuitive mathematical model.
2. The method also yields the eigenvector or mode shape and generalized mass corresponding to each natural frequency.
3. The accuracy of the dynamic response of a structure using impedance-type experimental data is not dependent on the accuracy of the test measurements, provided the data is within the state of the measurement art.
4. The mass matrix assumed for an intuitive mathematical model should be fully populated to yield accurate dynamic response results.
5. To insure minimum information loss in the inversion of mobility matrices, the averaging of mobility test data should be used in practice.
6. There is an upper limit to the size of a physically meaningful reduced complete model yielding minimum loss of information digits. The present report indicates the maximum to be a model of approximately 15 degrees of freedom.

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APPENDIX
COMPUTER PROGRAM DESCRIPTION

Note: All integer variables must be right justified with no decimal point.

Tape, Card Reader and Printer Assignments.

- 1 Card Reader
- 3 Printer
- 9 Contains influence coefficient matrix for use in XACT.
- 10 Tape assignment in XACT program. Contains mobility data for all degrees of freedom, with no error for specified frequencies for use in INXACT program.
- 4 11 Tape assignment in INXACT program. Contains mobility data with reduced stations and error (i.e., simulated test data) for use in program IDENFRE.

All input data must be in the following units.

Mass - lb-sec²/in.
Stiffness - lb/in.
Frequencies - Hz

PROGRAM XACT

| Card | 1 | Columns | 1 | IC | Program Control |
|-----------------|---|---------|---------------------------------|---------------------|--|
| | | | | IC \neq 0 | End Program |
| | | | | IC = 0 | Continue Program |
| | | | | HEAD | Case Description |
| Card | 2 | Columns | 1-10 11-20 21-30 31-40 | ND G NC NK | Number of Degrees of Freedom (≤ 20) Structural Damping Coefficient Number of Modes to be Obtained From Matrix Product [C][M]. If NC = 0, K is not inverted. Number of Modes to be Obtained from Matrix Product $[K]^{-1}[M]$. |
| Card(s) | 3 | | | M | Mass Matrix. (8E10.0 Format). For full symmetric matrix load lower triangular matrix only starting each row on a new card and ending with the diagonal element. Use as many cards as necessary. For a diagonal mass matrix, load one blank card followed by cards containing diagonal elements in sequence (8E10.0 Format). |
| Card(s) | 4 | | | K | For direct loading of K matrix from cards, proceed as for M matrix as described above. |
| C Matrix Option | | | | C | To load C matrix from TAPE 9, load one blank card. This will read C matrix from TAPE 9. Unformatted record contains heading (20 words, first character blank); NX (order of matrix). Force deflection influence coefficient matrix. |

| | | | | | |
|------------------------------|---|---------|-------------|-----------|--|
| Card | 5 | Columns | 1-5 6-10 | NF IP1 | Number of Frequencies Used (< 100) Print Control of Data Written on TAPE 10. IP1 = 0 No Printed Output Except List of Frequencies IP1 = 1 Print Full Mobility Matrix, Real and Imaginary at Each Frequency IP1 = 2 Print Only Diagonal Elements and Row of Mobility Matrix, Real and Imaginary at Each Frequency |
| Card(s) Omit if NF = 0 | 6 | Columns | 11-15 | IP2 | Control on Printed Output IP2 = 0 Same as Written on Tape Above, Complex Velocity Mobility Matrix at Each Frequency IP2 = 1 Print Acceleration Amplitude and Phase Angle This is the row to be printed when IP2 = 2. If NRØW = 0 then only diagonal (driving point) elements are printed as output. |
| Card | 7 | Columns | 16-20 | NRØW | Frequencies in Hertz. 10 Columns Per Value, 8 Values Per Card (100 Maximum). Format (8F10.2) |
| Card | 8 | Column | | | Frequency sweep control. This card is the same as Card 5 except that TAPE 10 is not written. To get response data with no tape use a blank card for Card 5 followed by Card 6. To generate TAPE 10 and print no other response data follow Card 5 by one blank card for Card 6. Both options indicated by Card 5 and Card 6 may be used simultaneously. For termination of Case Use 1 in Column 1. Blank card indicates another case to follow, beginning with card 1 again. |

PROGRAM INXACT

| | | | | | |
|---------|---|---------|--|--------------------------------|---|
| Card | 1 | Column | 1 | IC HEADN | Program Control Case Description |
| Card | 2 | Columns | 1-10 11-20 21-30 31-40 71-80 | NR PCT PCTB PHE IZ | Number of Points Tested (Number of Degrees of Freedom of the Model) Random Error Applied to Amplitude, Uniform between - and + PCT* Element Amplitude. Bias Error Applied to Amplitude. PCTB* Element Amplitude. Random Error in Degrees Applied to Phase Angle. Uniform Between -PHE and +PHE. Random Number Seed. |
| Card | 3 | | | KEEP | Stations to be used in model. Card 3 is included only if NR < ND (From Program XACT). Five columns per value, maximum of 10 values per card (Format 10I5) |
| Card | 4 | Columns | 1-5 6-10 11-15 16-20 | NFR IP1 IP2 NRØW | Number of Frequencies to be Used (From TAPE 10, XACT Program) IF NFR = 0 all frequencies on TAPE 10 are to be used. Same Definitions as in XACT Program |
| Card(s) | 5 | | | INDX | Indices of Frequencies to be Used from TAPE 10 XACT Program. Indices must be in ascending order. Five columns per value, 16 values per card (Format 16I5). |

PROGRAM IDENTRE

| Card | 1 | Columns | 1 | IC | Program Control |
|---------|---|---------|---|------|--|
| | | | | | IC = 1 Full Program Output IC > 1 Terminate Program Case Description |
| Card | 2 | Column | 1 | NNØR | Control on Normalization of Mobility Matrices |
| Card(s) | 3 | | | INDX | Indices of the Frequencies on TAPE 11 From INXACT Program to be Used in Summation of Real Parts of Mobility Elements (NFR Frequencies. Must be in Ascending Order) Five Columns Per Value, 16 Values Per Card (Format 16I5) |
| Card(s) | 4 | | | INDX | Indices of Frequencies to be Used for PHI Iteration (MODE SHAPE). Same Number of Indices as the Number of Degrees of Freedom of the Model. Indices in Ascending Order. |
| Card(s) | 5 | | | IØM | Indices of Frequencies to be Used in Forming $Y_{i(\omega)}$ and $Z_{i(\omega)}$ in the Calculation of Generalized Mass and Natural Frequency (2* Number of Degrees of Freedom of the Model). Indices in Ascending Order. |

| | | | | | |
|---------|---|---------|-------|------|--|
| Card | 6 | Columns | 1-5 | NF | No. of Frequencies at Which Reidentification of Mobility Matrices is Calculated. |
| | | | 6-10 | IP1 | Print Control of Mobility Data IP1 = 0 No printed output except list of frequencies IP1 = 1 Full matrices printed IP1 = 2 Diagonal elements and row printed |
| | | | 11-15 | IP2 | IP2 = 0 Complex velocity mobilities printed IP2 = 1 Acceleration mobilities printed Amplitude in g's and phase in degrees |
| | | | | NROW | This is Row to be Printed when IP1 = 2. If Equal to Zero the Only Diagonal (Driving Point) Elements are Printed |
| | | | 16-20 | NN | Controls Type of Damping Used in Reidentification of Mobilities NN = 0 Use Scalar Structural Damping Coefficient *K Matrix NN = 1 Use Damping Matrix |
| Card(s) | 7 | | | HZ | Frequencies at Which Reidentification is Calculated Ten Columns Per Value, 8 Values Per Card (Format 8F10.0). |
| Card | 8 | Column | 1 | | A 2 in Column 1 Terminates Program Otherwise Return to Card 1 for Beginning of New Case |

COMPUTER PROGRAM FORTRAN LISTING

```

C      XACT XACT XACT XACT XACT XACT XACT XACT XACT XACT XACT XACT XACT XACT
INTEG HEAD(20),HEAD1(20),IT(20),ITK(20)
INTEGER HT(7)
REAL M(20,21),M(20,21),C(20,21),A(20,21),B(20,21),PHI(20,21),
A FRE(20),DUM(20),GM(20),MU(20,21),PHIK(20,21),FREK(20),GNK(20)
REAL MZ(100),ZK(20,21),Z1(20,21),YA(20,21),YI(20,21),DPR(100,20),
A DPI(100,20),TR(100,20),TI(100,20)
LOGICAL TORF,TAPE
DATA HT/'EXAC',TA DA',TA S',IMUL',ATED', TES',: /
10
C      100 READ (1,110) IC,HEAD
110 FORMAT (11,A3,19A4)
IF ((IC.NE.0) GO TO 700
READ (1,120) ND,G,NC,NK
120 FORMAT (110,F10.0 ,2I10)
ND1=ND-1
READ (1,130) M(1,1)
130 FORMAT (8E10.0)
IF (M(1,1).NE.0) GO TO 150
C      DIAGONAL MASS
DO 140 I=1,ND
DO 140 J=1,ND
140 M(I,J)=0
GO TO 170
C      FULL MASS MATRIX
150 DO 160 I=2,ND
160 READ (1,130) (M(I,J),J=1,I)
CALL SYM (M,ND)
170 READ (1,130) K(1,1)
IF (K(1,1).EQ.0) GO TO 190
C      K INPUT
DO 180 I=2,ND
180 READ (1,130) (K(I,J),J=1,I)
CALL SYM (K,ND)
GO TO 230
C      C FROM TAPE
190 READ (9) HEAD1,NX,(C(I,J),I=1,NX),J=1,NK)
IF (ND.EQ.NX) GO TO 210
WRITE (3,200) HEAD,HEAD1
200 FORMAT (10X,A3,19A4//T5,*C MATRIX WRONG SIZE>//T5,*TAPE HEADING*,
A 10X,1H'A3,19A4,1H*)
CALL EXIT
C      INVERT AND SYMMETRIZE C
210 CALL INVR (C,ND,K)
DO 220 I=1,ND1
I1=I+1
DO 220 J=I1,ND
K(I,J)=(K(I,J)+K(J,I))/2.0
220 K(J,I)=K(I,J)
C      INVERT K
230 IF (NX.EQ.0.AND.NC.NE.0) CALL INVR (K,ND,C)
C      SUM K ROWS
240 DO 250 I=1,ND
1XCT 1
1XCT 2
1XCT 3
1XCT 4
1XCT 5
1XCT 6
1XCT 7
1XCT 8
1XCT 9
1XCT 10
1XCT 11
1XCT 12
1XCT 13
1XCT 14
1XCT 15
1XCT 16
1XCT 17
1XCT 18
1XCT 19
1XCT 20
1XCT 21
1XCT 22
2XCT 23
2XCT 24
1XCT 25
1XCT 26
1XCT 27
1XCT 28
1XCT 29
1XCT 30
1XCT 31
1XCT 32
1XCT 33
1XCT 34
1XCT 35
1XCT 36
1XCT 37
1XCT 38
1XCT 39
1XCT 40
1XCT 41
1XCT 42
1XCT 43
1XCT 44
1XCT 45
1XCT 46
1XCT 47
1XCT 48
2XCT 49
2XCT 50
2XCT 51
1XCT 52
1XCT 53
1XCT 54
1XCT 55

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```

K(I,21)=0
DO 250 J=1,ND
250 K(I,21)=K(I,21)+K(I,J) LIST INPUT
C
WRITE (3,260) HEAD,ND,G
260 FORMAT ('1',/T5,'20(' XACT ')/T5,15('*,5X,A3,19A4,5X,15('*,//
A 110,' DEGREES OF FREEDOM',10X,' STRUCTURAL DAMPING PARAMETER = ',
B F6.3//T50,' MASS MATRIX')
CALL MOUT2 (M,ND,ND)
IF (INX.EQ.0.AND.NC.EQ.0) GO TO 290
WRITE (3,270)
270 FORMAT ('1',/T45,' INFLUENCE COEFFICIENT MATRIX'//)
IF (INX.NE.0) WRITE (3,280) HEAD1
280 FORMAT ('*',/T5,' FROM TAPE'//T5,' TAPE HEADING',10X,1H'A3,19A4,1H'//)
CALL MOUT2 (C,ND,ND)
290 WRITE (3,300)
300 FORMAT ('1',/T50,' STIFFNESS MATRIX'//)
CALL MOUT2 (K,ND,ND)
WRITE (3,310) (K(I,21),I=1,ND)
310 FORMAT (/T50,' SPRINGS TO GROUND'//T10,1P10E12.4//)
IF (INC.EQ.0) GO TO 350
C CALC FREQ AND MODES COM FREQ IN HZ
DO 320 I=1,ND
DO 320 J=1,ND
320 R(I,J)=C(I,J)
DO 340 J=1,NC
CALL MOPY (B,M,ND,ND,ND,A)
CALL SITER (A,PHI,FRE,J,ND,(TN,PMAX)
FRE(J)=FRE(J)/6.283185
IT(J)=ITN
DO 330 I=1,ND
GM(I)=PHI(I,J)
GM(J)=GEN(DUM,M,ND)
CON=PMAX/GM(J)
DO 340 I=1,ND
DO 340 L=1,ND
340 B(I,L)=8(I,L)-DUM(I)*DUM(L)*CON
350 IF(INC.EQ.0) GJ TO 390
C FREQ AND MODES MU * C FREQ IN HZ
DO 360 I=1,ND
DO 360 J=1,ND
360 B(I,J)=K(I,J)
CALL INVR5 (M,ND,MU)
CALL MOPY (MU,B,ND,ND,ND,A)
DO 380 J=1,NC
CALL MOPY (B,M,ND,ND,ND,A)
CALL SITER (A,PHIK,FREK,J,ND,ITN,PMAX)
FREK(J)=1./FREK(J)/6.283185
ITK(J)=ITN
DO 370 I=1,ND
GMK(I)=PHIK(I,J)
GMK(J)=GEN(DUM,M,ND)
CON=PMAX/GMK(J)
DO 380 I=1,ND

```

```

1XCT 56
2XCT 57
XCT 58
XCT 59
XCT 60
XCT 61
XCT 62
XCT 63
XCT 64
XCT 65
XCT 66
XCT 67
XCT 68
XCT 69
XCT 70
XCT 71
XCT 72
XCT 73
XCT 74
XCT 75
XCT 76
XCT 77
1XCT 78
2XCT 79
2XCT 80
1XCT 81
1XCT 82
1XCT 83
1XCT 84
1XCT 85
2XCT 86
2XCT 87
1XCT 88
1XCT 89
2XCT 90
3XCT 91
3XCT 92
XCT 93
XCT 94
1XCT 95
2XCT 96
2XCT 97
XCT 98
XCT 99
XCT 100
1XCT 101
1XCT 102
1XCT 103
1XCT 104
1XCT 105
2XCT 106
2XCT 107
1XCT 108
1XCT 109
2XCT 110

```

```

380 B(I,L)=B(I,L)-DUM(I)*DUM(L)*CON
390 IF (NC.EQ.0) GO TO 430
C
WRITE (3,400)
400 FORMAT (1,'//745,'NORMAL MODES FROM C MATRIX'//)
CALL MOUT2 (PHI,ND,NC)
WRITE (3,410) (FRE(I),I=1,NC)
410 FORMAT (//745,'FREQUENCIES - HZ'//(10,10F12.6))
WRITE (3,420) (GM(I),I=1,NC)
420 FORMAT (//745,'GENERALIZED MASS'//(10,10F12.6))
430 IF (NK.EQ.0) GO TO 450
WRITE (3,440)
440 FORMAT (1,'//745,'NORMAL MODES FROM K MATRIX'//)
CALL MOUT2 (PHIK,ND,NK)
WRITE (3,410) (FREK(I),I=1,NK)
WRITE (3,420) (GMK(I),I=1,NK)
C
READ TAPE CONTROLS
450 TAPE=.TRUE.
460 READ (1,470) NF,IP1,IP2,NROW
470 FORMAT (4I5)
IF (.NOT.TAPE.AND.IP1.EQ.0) GO TO 100
IF (NF.EQ.0) GO TO 690
TORF=NROW-GT.0.AND.NROW.LE.ND
READ (1,130) (HZ(I),I=1,NF)
C
IF (TAPE) WRITE (10) HT,HEAD,NF,ND,(HZ(I),I=1,NF)
DO 570 L=1,NF
CALL MOB (M,K,G,ND,HZ(L),ZR,ZI,YR,YI)
IF (TAPE) WRITE (10) HZ(L),((YR(I,J),YI(I,J)),I=1,ND),J=1,ND)
IF (IP1-1) 570,480,550
480 IF (IP2.NE.0) CALL MATAMP (HZ(L),YR,YI,ND)
490 FORMAT (1,'T10,'COMPLEX MOBILITY WRITTEN ON TAPE'//)
IF (.NOT.TAPE) WRITE (3,540)
IF (IP2.NE.0) GO TO 510
WRITE (3,500) HZ(L)
500 FORMAT (7F40,'REAL MOBILITY, IMAGINARY MOBILITY FREQ =*F10.2,
A * HERTZ'//)
GO TO 530
510 WRITE (3,520) HZ(L)
520 FORMAT (7F40,'ACCELERATION AMPLITUDE IN G'S, PHASE IN DEG. FREQ
A =*F10.2,' HERTZ'//)
530 CALL MOUT2 (YR,ND,ND)
WRITE (3,540)
540 FORMAT (1,'//)
GO TO 570
550 DO 560 I=1,ND
DPR(I,1)=YR(I,1)
DPI(I,1)=YI(I,1)
IF (.NOT.TORF) GO TO 560
TR(I,1)=YR(NROW,I)
560 T(I,1)=YI(NROW,I)
570 CONTINUE

```

```

3XCT 111
3XCT 112
XCT 113
XCT 114
XCT 115
XCT 116
XCT 117
XCT 118
XCT 119
XCT 120
XCT 121
XCT 122
XCT 123
XCT 124
XCT 125
XCT 126
XCT 127
XCT 128
XCT 129
XCT 130
XCT 131
XCT 132
XCT 133
XCT 134
XCT 135
XCT 136
XCT 137
XCT 138
XCT 139
XCT 140
XCT 141
XCT 142
XCT 143
XCT 144
XCT 145
XCT 146
XCT 147
XCT 148
XCT 149
XCT 150
XCT 151
XCT 152
XCT 153
XCT 154
XCT 155
XCT 156
XCT 157
XCT 158
2XCT 159
2XCT 160
2XCT 161
2XCT 162
2XCT 163
2XCT 164
XCT 165

```

```

IF (IP1-1) 580,690,600
580 WRITE (3,590) (HZ(1),I-1,NF)
590 FORMAT (//T10,'MOBILITY MATRICES AT THE FOLLOWING FREQUENCIES (HZ
A) HAVE BEEN WRITTEN ON TAPE'//(T10,10F12.6))
GO TO 690
600 IF(IP2.NE.1) GO TO 620
CALL AMP (HZ,DPR,DPI,NF,ND)
IF(TORF) CALL AMP (HZ,TR,TI,NF,ND)
WRITE (3,610)
610 FORMAT ('1.T*0,'DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN
DEGREES'//)
GO TO 640
620 WRITE (3,630)
630 FORMAT ('1.T*0,'DRIVING POINT MOBILITY, REAL AND IMAGINARY'//)
640 CALL YOUT (HZ,DPR,NF,ND,0)
WRITE (3,640)
CALL YOUT (HZ,DPI,NF,ND,IP2)
IF(.NOT.TORF) GO TO 690
IF (IP2.NE.1) GO TO 660
WRITE (3,650) NROW
650 FORMAT ('1.T30,'TRANSFER RESPONSE, ROW '15,' AMP IN G'S AND PHAS
AE IN DEG'//)
GO TO 680
660 WRITE (3,670) NROW
670 FORMAT ('1.T30,'TRANSFER MOBILITY, ROW '15,' REAL AND IMAG'//)
680 CALL YOUT (HZ,TR,NF,ND,0)
WRITE (3,680)
CALL YOUT (HZ,TI,NF,ND,IP2)
690 IF (.NOT.TAPE) GO TO 100
TAPE = .FALSE.
GO TO 460
700 REWIND 10
CALL EXIT
END

```

```

XCT 166
XCT 167
XCT 168
XCT 169
XCT 170
XCT 171
XCT 172
XCT 173
XCT 174
XCT 175
XCT 176
XCT 177
XCT 178
XCT 179
XCT 180
XCT 181
XCT 182
XCT 183
XCT 184
XCT 185
XCT 186
XCT 187
XCT 188
XCT 189
XCT 190
XCT 191
XCT 192
XCT 193
XCT 194
XCT 195
XCT 196
XCT 197
XCT 198
XCT 199

```



```
1  
2  
3  
4  
5  
6  
7  
8  
9  
10
```

```
SUBROUTINE SYN (A,N)  
  FORMS SYMMETRIC MATRIX FROM LOWER TRIANGLE  
  REAL A(20,21)  
  N1=N-1  
  DO 100 I=1,N1  
    I1=I+1  
    DO 100 J=I1,N  
      100 A(I,J)=A(J,I)  
  RETURN  
  END
```

C

```

SUBROUTINE MOUTZ (A,M,N)
  REAL A(20,21)
  ID=MNO(N,10)
  WRITE (3,100) (I,I=1,10)
100 FORMAT (7F5,10I12)
  WRITE (3,100)
  DO 110 I=1,M
110 WRITE (3,120) I,(A(I,J),J=1,10)
120 FORMAT (15,5X,1P10E12.4)
  IF (10-N) 130,150,150
130 WRITE (3,100) (I,I=11,N)
  WRITE (3,100)
  DO 140 I=1,M
140 WRITE (3,120) I,(A(I,J),J=11,N)
150 RETURN
  END

```

```

MOT 1
MOT 2
MOT 3
MOT 4
MOT 5
MOT 6
MOT 7
MOT 8
MOT 9
MOT 10
MOT 11
MOT 12
MOT 13
MOT 14
MOT 15
MOT 16

```

```

C
C
C
FUNCTION GEN (FUN,A,N)
      GEN = FJNTRANS) * A * FJN
      DIMENSION A(20,21),FUN(20)
      GEN=0
      DO 110 I=1,N
      DUM=0
      DO 100 J=1,N
      DUM=DUM+A(I,J)*FUN(J)
      100 GEN=GEN+DUM*FUN(I)
      RETURN
      END
      GEN 1
      GEN 2
      GEN 3
      GEN 4
      GEN 5
      GEN 6
      1GEN 7
      1GEN 8
      2GEN 9
      1GEN 10
      1GEN 11
      GEN 12
      GEN 13

```

```

C
SUBROUTINE INVR (B,N,A)
A = INVERSE OF B      8 UNDISTURBED
DIMENSION A(20,21),D(20,21),IROW(21),ICOL(21),B(20,21)
DO 100 I=1,N
DO 100 J=1,N
100 A(I,J)=B(I,J)
M=N+1
DO 110 I=1,N
IROW(I)=I
ICOL(I)=I
DO 260 K=1,N
AMAX= A(K,K)
DO 130 I=K,N
DO 130 J=K,N
IF(ABS( A(I,J) )-ABS(AMAX))136,120,120
120 AMAX= A(I,J)
IC=I
JC=J
130 CONTINUE
KI=ICOL(K)
ICOL(K)=ICOL(IC)
ICOL(IC)=KI
KI=IROW(K)
IROW(K)=IROW(JC)
IROW(JC)=KI
IF(AMAX) 160,140,160
140 WRITE (3,150)
150 FORMAT(' SOLUTION OF EXISTING MATRIX NOT POSSIBLE')
GO TO 330
160 DO 170 J=1,N
E=A(K,J)
A(K,J)=A(IC,J)
170 A(IC,J)=E
DO 180 I=1,N
E=A(I,K)
A(I,K)=A(I,JC)
180 A(I,JC)=E
DO 210 I=1,N
IF(I-K) 200,190,200
190 A(I,M)=1.
GO TO 210
200 A(I,M)=0.
210 CONTINUE
PVT=A(K,K)
DO 220 J=L,M
220 A(K,J)=A(K,J)/PVT
DO 250 I=1,N
IF(I-K)230,250,230
230 AMULT=A(I,K)
DO 240 J=L,M
240 A(I,J)=A(I,J)-AMULT*A(K,J)
250 CONTINUE
DO 260 I=1,N
260 A(I,K)=A(I,M)
DO 290 I=1,N

```

```

INV 1
INV 2
INV 3
INV 4
INV 5
INV 6
INV 7
INV 8
INV 9
INV 10
INV 11
INV 12
INV 13
INV 14
INV 15
INV 16
INV 17
INV 18
INV 19
INV 20
INV 21
INV 22
INV 23
INV 24
INV 25
INV 26
INV 27
INV 28
INV 29
INV 30
INV 31
INV 32
INV 33
INV 34
INV 35
INV 36
INV 37
INV 38
INV 39
INV 40
INV 41
INV 42
INV 43
INV 44
INV 45
INV 46
INV 47
INV 48
INV 49
INV 50
INV 51
INV 52
INV 53
INV 54
INV 55

```

```

00 270 L=1,N
IF(IRO(I))-L)2 70,280,27C
270 CONTINUE
280 DO 290 J=1,M
290 D(L,J)=A(I,J)
DO 320 J=1,N
DO 300 L=1,N
IF(ICOL(J)-L) 300,310,300
300 CONTINUE
310 DO 320 I=1,N
320 A(I,L)=D(I,J)
330 RETURN
END

```

```

2 INV 56
2 INV 57
2 INV 58
2 INV 59
2 INV 60
1 INV 61
2 INV 62
2 INV 63
2 INV 64
2 INV 65
2 INV 66
1 INV 67
1 INV 68

```

```

C
C
C
C
SUBROUTINE MPMY (A,B,N1,N2,N3,C)
      C = A * B
      A (N1 X N2)  B (N2 X N3)  C (N1 X N3)
      REAL A(20,21),B(20,21),C(20,21)
      DO 100 I=1,N1
      DO 100 J=1,N3
      C(I,J)=0.
      DO 100 K=1,N2
      100 C(I,J)=C(I,J)+A(I,K)*B(K,J)
      RETURN
      END
      MMY 1
      MMY 2
      MMY 3
      MMY 4
      MMY 5
      MMY 6
      1MMY 7
      2MMY 8
      3MMY 9
      4MMY 10
      MMY 11
      MMY 12
      MMY 13

```

```

C
C
C
C
SUBROUTINE CINV (A,B,N,C,D)
      C+I*D = INVERSE OF A+I*B      I=SQRT(-1)
      B ASSUMED NON SINGULAR
      REAL A(20,21),B(20,21),C(20,21),D(20,21),E(20,21)
      CALL INVSIB(N,C)
      CALL MPPY(C,A,N,N,N,E)
      CALL MPPY(A,E,N,N,N,C)
      DO 100 I=1,N
      DO 100 J=1,N
      CALL INVSIC(N,D)
      DO 110 I=1,N
      DO 110 J=1,N
      D(I,J)=-D(I,J)
      100 RETURN
      110 END
      END
CIN 1
CIN 2
CIN 3
CIN 4
CIN 5
CIN 6
CIN 7
CIN 8
CIN 9
CIN 10
1CIN 11
2CIN 12
2CIN 13
CIN 14
CIN 15
1CIN 16
2CIN 17
2CIN 18
CIN 19
CIN 20

```

```

C      SUBROUTINE MOB (M,K,G,N,OM,ZR,ZI,YR,YI)
C
C      CALCULATES COMPLEX IMPEDANCE AND MOBILITY
C      M IS SQUARE MASS MATRIX
C      K IS SQUARE STIFFNESS MATRIX
C      G IS SCALAR STRUCTURAL DAMPING
C      OM IS FREQUENCY IN HERTZ
C      N IS ORDER
C
C      IMPEDANCE IS ZR + I*ZI      ( I = SORT(-1) )
C      MOBILITY = YR + I*YI
C
C      ALL SQUARE MATRICES ARE DIMENSIONED (20,21)
C
C      USES C:NV, INVR, MPMY
C
C      REAL M(20,21),K(20,21),ZR(20,21),ZI(20,21),YR(20,21),YI(20,21)
C      OMR=OM*6.283185
C      CON=G/OMR
C      DO 100 I=1,N
C      DO 100 J=1,N
C      ZR(I,J)=CON*K(I,J)
C      ZI(I,J)=OMR*M(I,J)-K(I,J)/OMR
C      CALL CINV (ZR,ZI,N,YR,YI)
C      RETURN
C      END
100

```

```

MOB 1
MOB 2
MOB 3
MOB 4
MOB 5
MOB 6
MOB 7
MOB 8
MOB 9
MOB 10
MOB 11
MOB 12
MOB 13
MOB 14
MOB 15
MOB 16
MOB 17
MOB 18
MOB 19
MOB 20
MOB 21
MOB 22
MOB 23
MOB 24
MOB 25
MOB 26

```



```

SUBROUTINE SITER (A,PHI,FRE,J,ND,ITN,PMAX)
REAL A(20,21),PHI(20,21),FRE(20),DUM(20)
K=ND-J+1
ANK=3.14159*K/(ND-1)
AN=3.14159*J/(ND-1)
DO 100 I=1,ND
ANG=ANK*(I-1)
ANGK=ANK*(I-1)
100 PHI(I,J)=(SINIANG)+SIN(ANGK)+1.0)/5.0
ITN=0
PMO=100.
110 DO 120 I=1,ND
DUM(I)=0.
120 DUM(I)=DUM(I)+A(I,L)*PHI(L,J)
PMAX=0.
DO 130 I=1,ND
130 PMAX=AMAX1(PMAX,ABS(DUM(I)))
DU 140 I=1,ND
140 PHI(I,J)=DUM(I)/PMAX
IF(ABS(PMAX/PMO-1.0)-.000001) 160,160,150
150 ITN=ITN+1
PMO=PMAX
IF(ITN=100) 110,110,160
160 FRE(J)=1.0/SQRT(ABS(PMAX))
RETURN
END

```

```

SIT 1
SIT 2
SIT 3
SIT 4
SIT 5
SIT 6
SIT 7
SIT 8
SIT 9
SIT 10
SIT 11
SIT 12
SIT 13
SIT 14
SIT 15
SIT 16
SIT 17
SIT 18
SIT 19
SIT 20
SIT 21
SIT 22
SIT 23
SIT 24
SIT 25
SIT 26
SIT 27

```

```

SUBROUTINE YOUT (OMH,A,NINC,ND,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MINO(ND,10)
IL=MINO(NINC,50)
100 IL=MINO(NINC,50)
11=1
110 WRITE (3,120) (I,I=J1,ID)
120 FORMAT (T5,'HERTZ',16,9I12)
WRITE (3,130)
130 FORMAT (1X)
IF(NAMP) 140,140,170
140 DO 150 I=1,IL
150 WRITE(3,160) OMH(I),(A(I,J),J=J1,ID)
160 FORMAT (1X,F9.3,1P10E12.4)
GO TO 200
170 DO 180 I=1,IL
180 WRITE(3,190) OMH(I),(A(I,J),J=J1,ID)
190 FORMAT (1X,F9.3,10F12.2)
200 IF(IL=NINC) 210,230,230
210 WRITE (3,220)
220 FORMAT ('1././)
IL=51
IL=NINC
GO TO 110
230 IF(10=ND) 240,250,250
240 J1=1
ID=ND
WRITE (3,190)
GO TO 100
250 RETURN
END

```

```

YOT 1
YOT 2
YOT 3
YOT 4
YOT 5
YOT 6
YOT 7
YOT 8
YOT 9
YOT 10
YOT 11
YOT 12
YOT 13
YOT 14
YOT 15
YOT 16
YOT 17
YOT 18
YOT 19
YOT 20
YOT 21
YOT 22
YOT 23
YOT 24
YOT 25
YOT 26
YOT 27
YOT 28
YOT 29
YOT 30
YOT 31

```

```

C
C
C
C
C
SUBROUTINE MATAMP (OMH,A,B,NR)
      CONVERTS MOBILITY, A + I*08 IN VEL JMITS TO
      AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
      MATRICES ARE AT FREQUENCY OMH IN HERTZ

      DIMENSION A(20,21),B(20,21)
      OM=OMH*0.01626
      DO 210 I=1,NR
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R**2+C**2)*OM
      IF(C) 140,100,140
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
MAT 7
MAT 8
MAT 9
MAT 10
MAT 11
MAT 12
MAT 13
MAT 14
MAT 15
MAT 16
MAT 17
MAT 18
MAT 19
MAT 20
MAT 21
MAT 22
MAT 23
MAT 24
MAT 25
MAT 26
MAT 27
MAT 28
MAT 29
MAT 30
MAT 31
MAT 32
MAT 33
MAT 34
MAT 35

```

```

C
C
C
C
C
SUBROUTINE AMP (OMH,A,B,NINC,NR)
      CONVERTS A ← I*B IN VELOCITY UNITS TO
      AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
      EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ
      DIMENSION OMH(100),A(100,20),B(100,20)
      DO 210 I=1,NINC
      OM=OMH(I)*0.01626
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R*R+C*C)*OM
      100 IF(R) 140,100,140
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      150 IF(C) 150,150,180
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
AMP 1
AMP 2
AMP 3
AMP 4
AMP 5
AMP 6
AMP 7
AMP 8
1AMP 9
1AMP 10
2AMP 11
2AMP 12
2AMP 13
2AMP 14
2AMP 15
2AMP 16
2AMP 17
2AMP 18
2AMP 19
2AMP 20
2AMP 21
2AMP 22
2AMP 23
2AMP 24
2AMP 25
2AMP 26
2AMP 27
2AMP 28
2AMP 29
2AMP 30
2AMP 31
2AMP 32
2AMP 33
AMP 34
AMP 35

```



```

CALL MOUTZ(YI, NR, NR)
INFR=INFR+1
GO TO 290
260 J=INFR
DO 270 I=1, NR
DPR(J, I)=YR(I, I)
DPI(J, I)=YI(I, I)
IF (.NOT. TORF) GO TO 270
YR(J, I)=YR(NRDM, I)
270 YI(J, I)=YI(NRDM, I)
280 HZ(INFR)=HZ(LL)
INFR=INFR+1
IF (INFR.GT. NFR) GO TO 300
290 CONTINUE
300 IF (IP1.NE.2) GO TO 390
IF (IP2.NE.1) GO TO 320
CALL AMP (HZ, DPR, DPI, NFR, NR)
IF (TORF) CALL AMP (HZ, TR, TI, NFR, NR)
WRITE (3, 310)
310 FORMAT ('1'//T30, 'DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN
A DEG'//)
GO TO 340
320 WRITE (3, 330)
330 FORMAT ('1'//T30, 'DRIVING POINT MOBILITY, REAL AND IMAGINARY'//)
340 CALL YOUT (HZ, DPR, NFR, NR, 0)
WRITE (3, 250)
CALL YOUT (HZ, DPI, NFR, NR, IP2)
IF (.NOT. TORF) GO TO 390
IF (IP2.NE.1) GO TO 360
WRITE (3, 350) NRDM
350 FORMAT ('1'//T30, 'TRANSFER RESPONSE, ROW'15,' , AMP IN G'S AND PH
AASE IN DEG'//)
GO TO 380
360 WRITE (3, 370) NRDM
370 FORMAT ('1'//T30, 'TRANSFER MOBILITY, ROW'15,' REAL AND IMAG'//)
380 CALL YOUT (HZ, TR, NFR, NR, 0)
WRITE (3, 250)
CALL YOUT (HZ, TI, NFR, NR, IP2)
390 REWIND 10
REWIND 11
CALL EXIT
END

```

```

11XT 56
11XT 57
11XT 58
11XT 59
21XT 60
21XT 61
21XT 62
21XT 63
21XT 64
21XT 65
11XT 66
11XT 67
11XT 68
11XT 69
11XT 70
11XT 71
11XT 72
11XT 73
11XT 74
11XT 75
11XT 76
11XT 77
11XT 78
11XT 79
11XT 80
11XT 81
11XT 82
11XT 83
11XT 84
11XT 85
11XT 86
11XT 87
11XT 88
11XT 89
11XT 90
11XT 91
11XT 92
11XT 93
11XT 94
11XT 95
11XT 96
11XT 97

```

```

C
C
C
C
C
C
C
C
C
C
SUBROUTINE ERR (A,B,PCT,PCTB,PHE,N,IX)
    EACH ELEMENT OF A COMPLEX MATRIX, A * 100, IS MODIFIED TO
    INCLUDE A SMALL PHASE ERROR, PHE (DEG), A BIAS ERROR,
    PCTB (RATIO) ON AMPLITUDE, AND A UNIFORM RANDOM ERROR
    HAVING A +/- MAXIMUM OF PCT (RATIO) ON AMPLITUDE.
    THE PHASE ERROR IS ALSO RANDOMLY DISTRIBUTED
    THE RESULTING MATRIX IS SYMMETRIZED

    USES RANDU

    DIMENSION A(20,21),B(20,21)
    IF(PCT) 120,100,120
    100 IF(PCTB) 120,110,120
    110 IF(PHE) 120,140,120
    120 P=PHE/57.296
    DO 130 I=1,N
    DO 130 J=1,N
    CALL RANDU (IX,IY,YFL)
    IX=IY
    E=2.0*P*(YFL-0.5)
    A1=A(I,J)-E*B(I,J)
    B(I,J)=B(I,J)+E*A(I,J)
    A(I,J)=A1
    CALL RANDU (IX,IY,YFL)
    IX=IY
    E=1.0+2.0*PCT*(YFL-0.5)+PCTB
    A(I,J)=A(I,J)*E
    130 B(I,J)=B(I,J)*E
    140 N1=N-1
    DO 150 I=1,N1
    J1=I+1
    DO 150 J=J1,N
    A(I,J)=(A(I,J)+A(J,I))/2.0
    B(I,J)=(B(I,J)+B(J,I))/2.0
    B(J,I)=B(I,J)
    150 A(J,I)=A(I,J)
    160 RETURN
    ENDO

```

```

ERR 1
ERR 2
ERR 3
ERR 4
ERR 5
ERR 6
ERR 7
ERR 8
ERR 9
ERR 10
ERR 11
ERR 12
ERR 13
ERR 14
ERR 15
ERR 16
ERR 17
ERR 18
1ERR 18
ZERR 19
ZERR 20
ZERR 21
ZERR 22
ZERR 23
ZERR 24
ZERR 25
ZERR 26
ZERR 27
ZERR 28
ZERR 29
ZERR 30
ERR 31
1ERR 32
1ERR 33
ZERR 34
ZERR 35
ZERR 36
ZERR 37
ZERR 38
ERR 39
ERR 40

```

1
2
3
4
5
6
7
8
9

RAN
RAN
RAN
RAN
RAN
RAN
RAN
RAN
RAN

```
C  
SUBROUTINE RANDU (IX,IY,YFL)  
  THIS SUBROUTINE IS FROM SSP VERS. II  
  IY=IX*65539  
  IF(IY) 100,10,110  
100 IY=IY*2147483647+1  
110 YFL=IY  
  YFL=YFL*.4656613E-9  
  RETURN  
  END
```



```

C
C
C
SUBROUTINE MATAMP (OMH,A,B,NR)
  CONVERTS MOBILITY, A → I0B IN VEL UNITS TO
  AMP (IN A ) IN G'S AND PHASE (IN B ) IN DEG
  MATRICES ARE AT FREQUENCY OMH IN HERTZ

  DIMENSION A(20,21),B(20,21)
  OM=OMH*0.01626
  DO 210 I=1,NR
  DO 210 J=1,NR
  R=A(I,J)
  C=B(I,J)
  A(I,J)=SQRT(R*R+C*C)*OM
  100 IF(R) 140,100,140
  110 B(I,J)=270.
  GO TO 210
  120 B(I,J)=0
  GO TO 210
  130 B(I,J)=90.
  GO TO 210
  140 P=ATAN(ABS(R/C))*57.2958
  IF(C) 150,150,180
  150 IF(R) 160,160,170
  160 B(I,J)=180.+P
  GO TO 210
  170 B(I,J)=180.-P
  GO TO 210
  180 IF(R) 190,190,200
  190 B(I,J)=360.-P
  GO TO 210
  200 B(I,J)=P
  210 CONTINUE
  RETURN
  END
MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
1MAT 7
2MAT 8
2MAT 9
2MAT 10
2MAT 11
2MAT 12
2MAT 13
2MAT 14
2MAT 15
2MAT 16
2MAT 17
2MAT 18
2MAT 19
2MAT 20
2MAT 21
2MAT 22
2MAT 23
2MAT 24
2MAT 25
2MAT 26
2MAT 27
2MAT 28
2MAT 29
2MAT 30
2MAT 31
MAT 32
MAT 33

```

```

SUBROUTINE AMP (OMH,A,B,NINC,NR)
C      CONVERTS A + I*B IN VELOCITY UNITS TO
C      AMP (IN A ) IN G'S AND PHASE ( IN B ) IN DEG
C      EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ
DIMENSION OMH(100),A(100,20),B(100,20)
DO 210 I=1,NINC
OM=OMH(I)*0.01626
DC 210 J=1,NR
R=A(I,J)
C=B(I,J)
A(I,J)=SQRT:(R*R+C*C)*OM
IF(C) 140,100,140
100 IF(R) 110,120,130
110 8(I,J)=270.
GO TO 210
120 8(I,J)=0
GO TO 210
130 8(I,J)=90.
GO TO 210
140 P=ATAN(ABS(R/C))*5/.2958
IF(C) 150,150,180
150 IF(R) 160,160,170
160 8(I,J)=180.+P
GO TO 210
170 8(I,J)=180.-P
GO TO 210
180 IF(R) 190,190,200
190 8(I,J)=360.-P
GO TO 210
200 8(I,J)=P
210 CONTINUE
RETURN
END

```

```

1 AMP
2 AMP
3 AMP
4 AMP
5 AMP
6 1AMP
7 1AMP
8 2AMP
9 2AMP
10 2AMP
11 2AMP
12 2AMP
13 2AMP
14 2AMP
15 2AMP
16 2AMP
17 2AMP
18 2AMP
19 2AMP
20 2AMP
21 2AMP
22 2AMP
23 2AMP
24 2AMP
25 2AMP
26 2AMP
27 2AMP
28 2AMP
29 2AMP
30 2AMP
31 2AMP
32 AMP
33 AMP

```

```
SUBROUTINE RED (A,B,NO,NR,KEEP )
INTEGER KEEP (20)
REAL A(20,21) , B(20,21)
DO 100 I=1,NR
DO 100 J=1,NO
A(I,J) = A(KEEP(I),KEEP(J))
100 B(I,J) = B(KEEP(I),KEEP(J))
RETURN
END
```

RED 1
RED 2
RED 3
1RED 4
2RED 5
2RED 6
2RED 7
RED 8
RED 9

```

SUBROUTINE YOUT (OM,A,NINC,ND,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MINO(MD,10)
IL=MINO(NINC,50)
I1=1
100 WRITE (3,120) (I,I-J1,10)
120 FORMAT (15,'HERTZ',16,9(112)
WRITE (3,130)
130 FORMAT (1X)
IF(NAMP) I40,I40,170
140 DO 150 I=I1,IL
150 WRITE(3,160) OMH(I),(A(I,J),J=J1,10)
160 FORMAT (1X,F9.3,1P10E12.4)
GO TO 200
170 DO 180 I=I1,IL
180 WRITE(3,190) OMH(I),(A(I,J),J=J1,10)
190 FORMAT (1X,F9.3,10E12.2)
200 IF(IL-NINC) 210,230,230
210 WRITE (3,220)
220 FORMAT ('1'//)
I1=51
IL=NINC
GO TO 110
230 IF(ID-ND) 240,250,250
240 J1=11
ID=ND
WRITE (3,190)
GO TO 100
250 RETURN
END
YOUT 1
YOUT 2
YOUT 3
YOUT 4
YOUT 5
YOUT 6
YOUT 7
YOUT 8
YOUT 9
YOUT 10
YOUT 11
YOUT 12
YOUT 13
YOUT 14
YOUT 15
YOUT 16
YOUT 17
YOUT 18
YOUT 19
YOUT 20
YOUT 21
YOUT 22
YOUT 23
YOUT 24
YOUT 25
YOUT 26
YOUT 27
YOUT 28
YOUT 29
YOUT 30
YOUT 31

```

```

SUBROUTINE MOUTZ (A,M,N)
REAL A(20,21)
ID=MIND(N,10)
WRITE (3,100) (I,I=1,10)
100 FORMAT (/Y5,10I12)
WRITE (3,100)
DO 110 I=1,M
110 WRITE (3,120) I,(A(I,J),J=1,10)
120 FORMAT (15,5X,1P10E12.4)
IF (10-N) 130,150,150
130 WRITE (3,100) (I,I=11,N)
WRITE (3,100)
DO 140 I=1,M
140 WRITE (3,120) I,(A(I,J),J=11,N)
150 RETURN
END

```

```

MOT 1
MOT 2
MOT 3
MOT 4
MOT 5
MOT 6
MOT 7
MOT 8
MOT 9
MOT 10
MOT 11
MOT 12
MOT 13
MOT 14
MOT 15
MOT 16

```

```

C      IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE
INTEGER HEAD(20),HEADN(20),HEAD(20),HTN(7),INDX(20),IT(20),
A  IOM(20,2),ITP(20)
REAL HZ(100),YRS(20,21),YR(20,21),YI(20,21),YRSIN(20,21),DUM(20),
A  PHI(20,21),GAMMA(20,21),OM(20,21),YRSTAR(20,21),YISTAR(20,21),
S  ZRSTAR(20,21),ZISTAR(20,21),GM(20),OMEGA(20),GK(20),GT(20)
REAL DUM1(20),GAM1(20,21),FACT(20)
LOGICAL EXCD
100 READ (1,110) IC,HEAD1
    IF(IC.GT.1) CALL EXIT
    READ (1,110) NNOR
110 FORMAT (11,A3,19A4)
    READ (11) HTN,HEAD,HEADN,NFR,NR,(HZ(1),I=1,NFR)
    WRITE (3,120) HEAD,HTN,HEAD,HEADN,NR,(HZ(1),I=1,NFR)
120 FORMAT ('1',/T5,12(' IDENTRE ',/T25,A3,19A4//T10,'TAPE HEADING',/
A  T25:7A4/2(T25,A3,19A4/),T25,'ORDER OF MATRICES =',T4/T25,'FREQUENC
    BIES ON TAPE',/T10,10F10.2))
    READ (1,130) (INDX(I),I=1,NFR)
    WRITE (3,140) (HZ(INDX(I)),I=1,NFR)
130 FORMAT (11C15)
140 FORMAT (/T25,'FIRST PASS FREQUENCIES',/(T10,10F10.2))
GO TO (150,170,190,210),NNOR
150 WRITE (3,160) FREQ
160 FORMAT ('  NORMALIZATION OF REAL MOBILITY BY MAX ABS(YR) ')
    GO TO 210
170 WRITE (3,180)
180 FORMAT ('  NORMALIZATION OF REAL MOBILITY BY RMS OF YR. ')
    GO TO 210
190 WRITE (3,200)
200 FORMAT ('  SUMMATION OF ACCELERATION MOBILITIES ')
    SUMM REAL PARTS AND INVERT (FIRST PASS)
C      DO 220 I=1,NR
    DO 220 J=1,NR
    INFR=1
    DO 310 L=1,NFR
    READ (11) FREQ,(YR(I,J),YI(I,J),I=1,NR),J=1,NR)
    NORMALIZATION OF MOBILITY MATRICES IF NNOR=1,2
    IF (L.NE.INDX(INFR)) GO TO 310
    GO TO (230,250,270,290),NNOR
230 CALL YRNRM (YR,NR)
    WRITE (3,240) FREQ
240 FORMAT ('1',/ REAL MOB NORMALIZED ON YR (MAX) FREQ=',F8.3,'HZ' )
    GO TO 290
250 CALL YRFRMS ( YR,NR )
    WRITE (3,260) FREQ
260 FORMAT ('1',/ REAL MOB NORMALIZED ON RMS OF YR FREQ=',F8.3,'HZ' )
    GO TO 290
270 CALL YRFRREQ (YR,FREQ ,NR)
    WRITE (3,280) FREQ
280 FORMAT ('1',/ ACCELERATION MOBILITY FREQ=',F8.3,'HZ' )
290 CALL MOUT2 ( YR,NR,NR )
    DO 300 I=1,NR
    DO 300 J=1,NR
300 YRS(I,J)=YRS(I,J)+YR(I,J)
IDN 1
IDN 2
IDN 3
IDN 4
IDN 5
IDN 6
IDN 7
IDN 8
IDN 9
IDN 10
IDN 11
IDN 12
IDN 13
IDN 14
IDN 15
IDN 16
IDN 17
IDN 18
IDN 19
IDN 20
IDN 21
IDN 22
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IDN 44
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IDN 49
IDN 50
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IDN 52
IDN 53
IDN 54
IDN 55

```

11DN 56
 11DN 57
 11DN 58
 11DN 59
 11DN 60
 11DN 61
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 11DN 68
 11DN 69
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 21DN 91
 21DN 92
 11DN 93
 11DN 94
 11DN 95
 11DN 96
 11DN 97
 11DN 98
 11DN 99
 11DN 100
 11DN 101
 11DN 102
 11DN 103
 11DN 104
 11DN 105
 11DN 106
 11DN 107
 11DN 108
 11DN 109
 21DN 110

```

IMR=IMR+1
IF (IMR.GT.NR) GO TO 320
310 CONTINUE
320 CALL INVR (YRS,NR,YRSIN)
    READ (11)
    IF (IC.EQ.0) GO TO 350
    WRITE (3,330)
330 FORMAT ('1',//T30,'SUM OF REAL MOBILITIES'//)
    CALL MOUT2 (YRS,NR,NR)
    WRITE (3,340)
340 FORMAT ('1',//T30,'INVERSE OF SUM OF REAL MOB'//)
    CALL MOUT2 (YRSIN,NR,NR)
    ITERATE FOR PHI (SECOND PASS)
350 READ (1,130) (INDX(I),I=1,NR)
    WRITE (3,360) (HZ(INDX(I)),I=1,NR)
360 FORMAT (//T25,'SECOND PASS FREQUENCIES'//((T10,10F10.2))
    IMR=1
    DO 380 L=1,NR
    READ (11) FREQ((YR(I),J),YI(I,J),I=1,NR),J=1,NR)
    IF (L.NE.INDX(IMR)) GO TO 380
    CALL MITER (YR,YRSIN,NR,.0001,99,DUM,VAL,ITN)
    ITP(IMR)=ITN
    CALL MITER (YRSIN,YR,NR,.0001,99,DUM1,VAL,ITN)
    ITP(IMR)=ITN
    DO 370 I=1,NR
    GAM(I,IMR)=SUM(I)
370 PHI(I,IMR)=DUM(I)
    IMR=IMR+1
    IF (IMR.GT.NR) GO TO 390
380 CONTINUE
390 DO 420 I=1,NR
    SUM=0.
    DO 400 J=1,NR
    SUM=SUM+GAM(I,J,IMR)*PHI(J,I)
    DO 410 J=1,NR
    GAM(I,J,I)=GAM(I,J,I)/SUM
420 CONTINUE
    WRITE (3,430)
430 FORMAT ('1',//T40,'ITERATED PHI'//)
    CALL MOUT2 (PHI,NR,NR)
    WRITE (3,440) (ITP(I),I=1,NR)
440 FORMAT (//T40,'ITERATIONS'//((T5,10I12))
    WRITE (3,450)
450 FORMAT ('1',//T40,'ITERATED GAMMA'//)
    CALL MOUT2 (GAM,NR,NR)
    WRITE (3,460) (IT(I),I=1,NR)
    EXCD = .FALSE.
    DO 460 I=1,NR
    IF (IT(I).GT.99) EXCD=.TRUE.
460 CONTINUE
    IF (EXCD) WRITE (3,470)
470 FORMAT (//T10,'*** WARNING - ITERATION NOT CONVERGED ***')
    DO 480 I=1,NR
    DO 480 J=1,NR
  
```

```

480 YI(I,J)=PHI(J,I)
CALL INVR5 (YI, NR, GAMMA)
WRITE (3,490)
490 FORMAT (///T40,'GAMMA = PHI INVERSE TRANSPOSE'//)
CALL MOUT2 (GAMMA, NR, NR)
      READ THIRD PASS FREQ
C
500 REWIND 11
READ (11)
READ (1,130) (OM(I,1), IOM(I,2), I=1, NR)
WRITE (3,510) (HZ(IOM(I,1)), HZ(IOM(I,2)), I=1, NR)
510 FORMAT ('1'//T25,'THIRD PASS FREQUENCIES'// (T10, 10F10.2))
      FORM ALL Y STAR
C
      I12=1
      INFR=1
      DD 550 L=1, NR
      READ (11) FREQ, (YR(I, J), YI(I, J), I=1, NR), J=1, NR)
      IF (L.NE.IOM(INFR, I12)) GO TO 550
      OM(INFR, I12)=FREQ
      IF (I12.EQ.2) GO TO 530
      DD 520 I=1, NR
      GAMMA(I, INFR)=GAMI(I, INFR)
520 DUM(I)=GAMMA(I, INFR)
530 YRSTAR(INFR, I12)=GEN(DUM, YR, NR)
      YISTAR(INFR, I12)=GEN(DUM, YI, NR)
      IF (I12.EQ.2) GO TO 540
      I12=2
      GO TO 550
540 I12=1
      INFR=INFR+1
      IF (INFR.GT. NR) GO TO 560
550 CONTINUE
      FORM Z STAR
C
560 DD 570 L=1, NR
      DD 570 LL=1, 2
      CON=YRSTAR(L, LL)*2+YISTAR(L, LL)*2
      ZRSTAR(L, LL)=YRSTAR(L, LL)/CON
570 ZISTAR(L, LL)=-YISTAR(L, LL)/CON
      WRITE (3,580)
580 FORMAT ('1', T40, 'YSTAR USING ITERATED GAMMA'//)
      WRITE (3,590)
590 FORMAT (1, T45, 'YSTAR (MODE)*T98, *ZSTAR (MODE)*T13, *MODE OM 1 REAL (OM 1) IMAG (OM 2) (OM 2) REAL (OM 2) IMAG (OM 1) (OM 2) (OM 2)*//)
      B (OM 1)
      WRITE (3,600) (I, OM(I, 1), YRSTAR(I, 1), YISTAR(I, 1), ZRSTAR(I, 1),
      A ZISTAR(I, 1), OM(I, 2), YRSTAR(I, 2), YISTAR(I, 2), ZRSTAR(I, 2),
      B ZISTAR(I, 2), I=1, NR)
600 FORMAT (15, OPF10.2, 1P4E24.4, /T18, OPF10.2, 1P4E24.4)
      IDENTIFY GEN MASS, NAT FREQ
C
      DD 610 I=1, NR
      GK(I)=(OM(I, 1)+ZISTAR(I, 1)-OM(I, 2)+ZISTAR(I, 2))/(OM(I, 1)+2-
      A OM(I, 2)+2)/6.283185
      OMEGA(I)=OM(I, 1)+OM(I, 2)*(OM(I, 2)+ZISTAR(I, 1)-OM(I, 1)+ZISTAR(I, 2))
      A / (OM(I, 1)+ZISTAR(I, 1)-OM(I, 2)+ZISTAR(I, 2))
      GK(I)=OMEGA(I)*GMI(I)*39.4784
      IF (OMEGA(I).GT.0) OMEGA(I)=SQRT(OMEGA(I))

```

```

2 IDN 111
IDN 112
IDN 113
EDN 114
EDN 115
IDN 116
IDN 117
IDN 118
IDN 119
IDN 120
IDN 121
IDN 122
IDN 123
IDN 124
IDN 125
IDN 126
IDN 127
IDN 128
IDN 129
IDN 130
2 IDN 131
2 IDN 132
IDN 133
IDN 134
IDN 135
IDN 136
IDN 137
IDN 138
IDN 139
IDN 140
IDN 141
IDN 142
IDN 143
IDN 144
2 IDN 145
2 IDN 146
2 IDN 147
IDN 148
IDN 149
IDN 150
IDN 151
IDN 152
IDN 153
IDN 154
IDN 155
IDN 156
IDN 157
IDN 158
IDN 159
IDN 160
IDN 161
IDN 162
IDN 163
IDN 164
IDN 165

```



```

G(I)=OM(I,1)*ZRSTAR(I,1)/OMEGA(I)*OMEGA(I)*CM(1)*96.283185)
610 CONTINUE
      WRITE (3,620) (I,CM(I),OMEGA(I),I*1,MR)
620 FORMAT ('1'//T40,'GENERALIZED MASSES AND NATURAL FREQUENCIES'//
A T50,'MODE GEN MASS',5X,'NAT FREQ'(F10.4,F15.5))
      REIDN (MR,CM,OMEGA,PHI,GAM1,GK,G)
      CALL REIDN (MR,CM,OMEGA,PHI)
      REWIND 11
      GO TO 100
      END
C
IDN 166
IDN 167
IDN 168
IDN 169
IDN 170
IDN 171
IDN 172
IDN 173
IDN 174
IDN 175

```

```

SUBROUTINE MITER (A,B,N,TOL,ITMAX,FUN,VAL,IT)
ITERATES ON A*B FOR DOMINANT EIGENFUNCTION (FUN)
AND EIGENVALUE (VAL).
N IS ORDER
TOL IS DECIMAL (.01 PERCENT) TOLERANCE ON VAL.
ITMAX IS MAX NO OF ITERATIONS.
IT IS NUMBER OF ITERATIONS PERFORMED.
A,B ARE SQUARE OF ORDER N (DIMENSIONED (20,21) ).
USES MPMY (A,B,N1,N2,N3,C)
REAL A(20,21),B(20,21),C(20,21),DUM(20),FUN(20)
CALL MPMY (A,B,N,N,N,C)
VALO=100.
IT=1
DO 100 I=1,N
FUN(I)=1.C
100 CALL MPMY (C,FUN,N,N,1,DUM)
VAL=DUM(1)
DO 130 I=2,N
IF (ABS(VAL)-ABS(DUM(I))) 120,130,130
120 VAL=DUM(I)
130 CONTINUE
DO 140 I=1,N
FUN(I)=DUM(I)/VAL
140 IF (ABS(VAL/VALO-1.0)-TOL) 160,160,150
150 IT=IT+1
VALO=VAL
IF (IT-ITMAX) 110,110,160
160 RETURN
END

```

```

MIT 1
MIT 2
MIT 3
MIT 4
MIT 5
MIT 6
MIT 7
MIT 8
MIT 9
MIT 10
MIT 11
MIT 12
MIT 13
MIT 14
MIT 15
MIT 16
MIT 17
MIT 18
MIT 19
MIT 20
MIT 21
MIT 22
MIT 23
MIT 24
MIT 25
MIT 26
MIT 27
MIT 28
MIT 29
MIT 30
MIT 31
MIT 32

```

```

SUBROUTINE MOUTZ (A,M,N)
REAL A(20,21)
ID=MINO(N,10)
WRITE (3,100) (I,I=1,ID)
100 FORMAT (/75,10I12)
WRITE (3,100)
DO 110 I=1,M
110 WRITE (3,120) I,(A(I,J),J=1,ID)
120 FORMAT (15,5X,1P10E12.4)
IF (ID=N) 130,150,150
130 WRITE (3,100) (I,I=11,M)
WRITE (3,100)
DO 140 I=1,M
140 WRITE (3,120) I,(A(I,J),J=11,N)
150 RETURN
END

```

```

MOT 1
MOT 2
MOT 3
MOT 4
MOT 5
MOT 6
MOT 7
MOT 8
MOT 9
MOT 10
MOT 11
MOT 12
MOT 13
MOT 14
MOT 15
MOT 16

```

```

GEN 1
GEN 2
GEN 3
GEN 4
GEN 5
1GEN 6
1GEN 7
2GEN 8
2GEN 9
1GEN 10
GEN 11
GEN 12

```

```

C
C
FUNCTION GEN (FUN,A,N) GEN = FJN(TRANS) * A * FJN
DIMENSION A(20,21),FUN(20)
GEN=0
DO 110 I=1,N
SUM=0
DO 100 J=1,N
100 DUM=DUM+A(I,J)*FUN(J)
110 GEN=GEN+DUM*FUN(I)
RETURN
END

```

```

C
SUBROUTINE INVRS (B,N,A)
A = INVERSE OF B      B UNDISTURBED
C
DIMENSION A(20,21),D(20,21),IROW(21),ICOL(21),M(20,21)
DO 100 I=1,N
DO 100 J=1,N
100 A(I,J)=B(I,J)
M=N+1
DO 110 I=1,N
IROW(I)=I
110 ICOL(I)=I
DO 260 K=1,M
AMAX= A(K,K)
DO 130 I=K,N
DO 130 J=K,N
IF(ABS( A(I,J)-ABS(AMAX)))130,120,120
120 AMAX= A(I,J)
IC=I
JC=J
130 CONTINUE
KI=ICOL(K)
ICOL(K)=ICOL(IC)
ICOL(IC)=KI
KI=IROW(K)
IROW(K)=IROW(JC)
IROW(JC)=KI
IF(AMAX) 160,140,160
140 WRITE (3,150)
150 FORMAT(' SOLUTION OF EXISTING MATRIX NOT POSSIBLE')
GO TO 330
160 DO 170 J=1,N
E=A(K,J)
A(K,J)=A(IC,J)
170 A(IC,J)=E
DO 180 I=1,N
E=A(I,K)
A(I,K)=A(I,JC)
180 A(I,JC)=E
DO 210 I=1,N
IF(I-K) 200,190,200
190 A(I,M)=1.
GO TO 210
200 A(I,M)=0.
210 CONTINUE
PVT=A(K,K)
DO 220 J=1,M
DO 220 J=1,M
220 A(K,J)=A(K,J)/PVT
DU 250 I=1,N
IF(I-K)230,250,230
230 AMULT=A(I,K)
DO 240 J=1,M
240 A(I,J)=A(I,J)-AMULT*A(K,J)
250 CONTINUE
DO 260 I=1,N
260 A(I,K)=A(I,M)

```

```

1 INV
2 INV
3 INV
4 INV
5 1INV
6 2INV
7 2INV
8 INV
9 1INV
10 1INV
11 1INV
12 1INV
13 1INV
14 2INV
15 3INV
16 3INV
17 3INV
18 3INV
19 3INV
20 3INV
21 1INV
22 1INV
23 1INV
24 1INV
25 1INV
26 1INV
27 1INV
28 1INV
29 1INV
30 1INV
31 2INV
32 2INV
33 2INV
34 2INV
35 2INV
36 2INV
37 2INV
38 2INV
39 2INV
40 2INV
41 2INV
42 2INV
43 2INV
44 2INV
45 1INV
46 2INV
47 2INV
48 2INV
49 2INV
50 2INV
51 3INV
52 3INV
53 2INV
54 2INV
55 2INV

```

```
DO 290 I=1,N
DO 270 L=1,N
IF(LKRM(I)-L)270,280,270
270 CONTINUE
280 DO 290 J=1,N
290 D(L,J)=A(I,J)
DO 300 L=1,N
IF(ICOL(J)-L) 300,310,300
300 CONTINUE
310 DO 320 I=1,N
320 A(L,I)=D(I,J)
330 RETURN
END
```

```
1 INV 56
2 INV 57
2 INV 58
2 INV 59
2 INV 60
2 INV 61
2 INV 62
2 INV 63
2 INV 64
2 INV 65
2 INV 66
2 INV 67
INV 68
INV 69
```

```

SUBROUTINE MNPY (A,B,N1,N2,N3,C)
C
C
C
      C = A * B
      A (N1 X N2)  B (N2 X N3)  C (N1 X N3)

      REAL A(20,21),B(20,21),C(20,21)
      DO 100 I=1,N1
      DO 100 J=1,N3
      C(I,J)=0.
      DO 100 K=1,N2
      100 C(I,J)=C(I,J)+A(I,K)*B(K,J)
      RETURN
      END
MNPY 1
MNPY 2
MNPY 3
MNPY 4
MNPY 5
MNPY 6
MNPY 7
MNPY 8
MNPY 9
MNPY 10
MNPY 11
MNPY 12
MNPY 13

```

```

1 YRN
2 YRN
3 YRN
4 1YRN
5 2YRN
6 2YRN
7 2YRN
8 2YRN
9 1YRN
10 2YRN
11 2YRN
12 YRN
13 YRN
14 YRN
15 YRN
16 YRN
17 YRN
18 1YRN
19 2YRN
20 2YRN
21 YRN
22 1YRN
23 2YRN
24 2YRN
25 YRN
26 YRN

```

```

SUBROUTINE YRNRM ( YR, NR )
DIMENSION YR(20,21)
VAL=YR(I,I)
DO 110 I=1, NR
DO 110 J=1, NR
IF( ABS(VAL)-ABS(YR(I,J))) 100,110,110
100 VAL=YR(I,J)
110 CONTINUE
DO 120 I=1, NR
DO 120 J=1, NR
120 YR(I,J)=YR(I,J)/ABS(VAL)
RETURN
END
SUBROUTINE YRRMS ( YR, NR )
YR NORMALIZATION BY RMS OF YR
DIMENSION YR(20,21)
RMS=0.
DO 130 I=1, NR
DO 130 J=1, NR
130 RMS=YR(I,J)*YR(I,J)+RMS
RMS=SQR(RMS/(NR*NR))
DO 140 I=1, NR
DO 140 J=1, NR
140 YR(I,J)=YR(I,J)/RMS
RETURN
END

```

C

| | | |
|----|------|----|
| 1 | MB2 | 1 |
| 2 | MB2 | 2 |
| 3 | MB2 | 3 |
| 4 | MB2 | 4 |
| 5 | MB2 | 5 |
| 6 | MB2 | 6 |
| 7 | MB2 | 7 |
| 8 | MB2 | 8 |
| 9 | MB2 | 9 |
| 10 | MB2 | 10 |
| 11 | MB2 | 11 |
| 12 | MB2 | 12 |
| 13 | MB2 | 13 |
| 14 | MB2 | 14 |
| 15 | MB2 | 15 |
| 16 | MB2 | 16 |
| 17 | MB2 | 17 |
| 18 | MB2 | 18 |
| 19 | MB2 | 19 |
| 20 | MB2 | 20 |
| 21 | MB2 | 21 |
| 22 | MB2 | 22 |
| 23 | MB2 | 23 |
| 24 | MB2 | 24 |
| 25 | MB2 | 25 |
| 26 | 1MB2 | 26 |
| 27 | 2MB2 | 27 |
| 28 | 2MB2 | 28 |
| 29 | 2MB2 | 29 |
| 30 | 2MB2 | 30 |
| 31 | 2MB2 | 31 |
| 32 | 2MB2 | 32 |
| 33 | MB2 | 33 |
| 34 | MB2 | 34 |
| 35 | MB2 | 35 |


```

SUBROUTINE MOB2(M,K,G,N,OM,ZR,ZI,YR,YI,D,IS)
  CALCULATES COMPLEX IMPEDANCE AND MOBILITY
  M IS SQUARE MASS MATRIX
  K IS SQUARE STIFFNESS MATRIX
  G IS SCALAR STRUCTURAL DAMPING
  D IS SCALAR DAMPING MATRIX
  OM IS FREQUENCY IN HERTZ
  N IS ORDER
  EITHER G OR D IS USED
  IF IS = 0 ZR = G*K/OMR
  IF IS = 1 ZR = D/OMR
  IMPEDANCE IS ZR + I*ZI      ( I = SQRT(-1) )
  MOBILITY = YR + I*YI
  ALL SQUARE MATRICES ARE DIMENSIONED (20,21)
  USES CINV, INVRS, NMPY
  REAL M(20,21),K(20,21),ZR(20,21),ZI(20,21),YR(20,21),YI(20,21)
  OMR=OM*6.283185
  IF (IS.EQ.0) CON=G/OMR
  DO 110 I=1,N
  DO 110 J=1,N
  ZR(I,J)=D(I,J)/OMR
  GO TO 110
  100 ZR(I,J)=CON*K(I,J)
  110 ZI(I,J)=OMR*M(I,J)-K(I,J)/OMR
  CALL CINV (ZR,ZI,N,YR,YI)
  RETURN
  END
  
```

```

C
C
C
C
SUBROUTINE CINV (A,B,M,C,D)
      C+I*D = INVERSE OF A+I*B      I=SQRT(-1)
      B ASSUMED NON SINGULAR
      REAL A(20,21),B(20,21),C(20,21),D(20,21),E(20,21)
      CALL INVRSEB,N,C
      CALL HMPY(C,A,N,M,N,E)
      CALL HMPY(A,E,N,N,N,C)
      DO 100 I=1,N
      DO 100 J=1,N
100  C(I,J)=C(I,J)+B(I,J)
      CALL INVRSC,N,D
      CALL HMPY(E,D,N,N,N,C)
      DO 110 I=1,N
      DO 110 J=1,N
110  D(I,J)=D(I,J)
      RETURN
      END
CIN 1
CIN 2
CIN 3
CIN 4
CIN 5
CIN 6
CIN 7
CIN 8
CIN 9
CIN 10
ICIN 11
2CIN 12
2CIN 13
CIN 14
CIN 15
ICIN 16
2CIN 17
2CIN 18
CIN 19
CIN 20

```

```
SUBROUTINE YRFREQ (YR,FREQ,NR )
DIMENSION YR(20,21)
DO 100 I=1,NR
DO 100 J=1,NR
100 YR(I,J)=YR(I,J)*FREQ
RETURN
END
```

```
YRF 1
YRF 2
1YRF 3
2YRF 4
2YRF 5
YRF 6
YRF 7
```

```

SUBROUTINE YOUT (OMH,A,NINC,NO,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MINO(MD,10)
IL=MINO(NINC,50)
100 J1=1
110 WRITE (3,120) (I,I=J1,10)
120 FORMAT (15,'HERTZ',16,9I12)
130 WRITE (3,130)
130 FORMAT (1X)
140 IF(NAMP) 140,140,170
140 DO 150 I=1,IL
150 WRITE(3,160) OMH(I),(A(I,J),J=J1,10)
160 FORMAT (1X,F9.3,1P10E12.4)
GO TO 200
170 DO 180 I=1,IL
180 WRITE(3,190) OMH(I),(A(I,J),J=J1,10)
190 FORMAT (1X,F9.3,10F12.2)
200 IF(IL=NINC) 210,230,230
210 WRITE (3,220)
220 FORMAT ('1'//)
I1=51
IL=NINC
GO TO 110
230 IF(ID=MD) 240,250,250
240 J1=11
ID=MD
WRITE (3,190)
GO TO 100
250 RETURN
END

```

```

YOUT 1
YOUT 2
YOUT 3
YOUT 4
YOUT 5
YOUT 6
YOUT 7
YOUT 8
YOUT 9
YOUT 10
YOUT 11
YOUT 12
YOUT 13
YOUT 14
YOUT 15
YOUT 16
YOUT 17
YOUT 18
YOUT 19
YOUT 20
YOUT 21
YOUT 22
YOUT 23
YOUT 24
YOUT 25
YOUT 26
YOUT 27
YOUT 28
YOUT 29
YOUT 30
YOUT 31

```

```

C
C
C
C
SUBROUTINE MATAMP (OMH,A,B,NR)
      CONVERTS MOBILITY, A ← JOB IN VEL UNITS TO
      AMP (IN A ) IN G'S AND PHASE (IN B ) IN DEG
      MATRICES ARE AT FREQUENCY OMH IN HERTZ

      DIMENSION A(20,21),B(20,21)
      OM=OMH*0.01626
      DO 210 I=1,NR
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SORT(R*H+C*C)*OM
      100 IF(R) 110,100,140
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      150 IF(C) 150,150,180
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
MAT 7
MAT 8
MAT 9
2MAT 10
2MAT 11
2MAT 12
2MAT 13
2MAT 14
2MAT 15
2MAT 16
2MAT 17
2MAT 18
2MAT 19
2MAT 20
2MAT 21
2MAT 22
2MAT 23
2MAT 24
2MAT 25
2MAT 26
2MAT 27
2MAT 28
2MAT 29
2MAT 30
2MAT 31
2MAT 32
2MAT 33
MAT 34
MAT 35

```

```

C
C
C
C
SUBROUTINE AMP (OMH,A,B,NINC,NR)
      CONVERTS A + I*B IN VELOCITY UNITS TO
      AMP (IN A) IN G*5 AND PHASE (IN B) IN DEG
      EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ

      DIMENSION OMH(100),A(100,20),B(100,20)
      DO 210 I=1,NINC
      OM=OMH(I)*0.01626
      DU 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R*R+C*C)*OM
      IF(C) 140,100,140
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
AMP 1
AMP 2
AMP 3
AMP 4
AMP 5
AMP 6
AMP 7
AMP 8
1AMP 9
2AMP 10
2AMP 11
2AMP 12
2AMP 13
2AMP 14
2AMP 15
2AMP 16
2AMP 17
2AMP 18
2AMP 19
2AMP 20
2AMP 21
2AMP 22
2AMP 23
2AMP 24
2AMP 25
2AMP 26
2AMP 27
2AMP 28
2AMP 29
2AMP 30
2AMP 31
2AMP 32
2AMP 33
AMP 34
AMP 35

```

```

SUBROUTINE REIDN (NR,GM,OM,PHI,GAM1,GK,G )
IDENTIFICATION OF MASS, STIFFNESS, DAMPING MATRICES
DIMENSION GM(20),OM(20),PHI(20,21),AM(20,21),AK(20,21)
DIMENSION GAM1(20,21),GK(20),AD(20,21),G(20)
DIMENSION C(20,21),U(20,21),CONA(20),AMG(20,21)
DIMENSION ZR(20,21),ZI(20,21),YR(20,21),YI(20,21),MZ(100)
DIMENSION DPR(100,20),DPI(100,20),YR(100,20),YI(100,20)
LOGICAL TORF
DO 100 I=1,NR
DO 100 J=1,NR
AD(I,J)=0.
AMG(I,J)=0.
UI(I,J)=0.
100 C(I,J)=0.
DO 120 I=1,NR
CONA(I)=1./GM(I)*OM(I)*OM(I)*39.4784
DO 110 J=1,NR
DO 110 K=1,NR
CALC=PHI(K,I)*PHI(J,I)
CAL=GAM1(K,I)*GAM1(J,I)
AD(K,J)=CAL*G(I)*GK(I)+AD(K,J)
AMG(K,J)=CAL*GM(I)+AMG(K,J)
U(K,J)=CAL/GM(I)+U(K,J)
110 C(K,J)=CAL*CONA(I)+C(K,J)
120 CONTINUE
CALL INVR5 (C,NR,AK)
CALL INVR5 (U,NR,AM)
WRITE (3,130)
CALL MOUT2 (AM,NR,NR)
130 FORMAT ('1',T50,'IDENTIFIED MASS MATRIX'//)
WRITE (3,140)
CALL MOUT2 (AK,NR,NR)
140 FORMAT ('1',T50,'IDENTIFIED STIFFNESS MATRIX'//)
WRITE (3,150)
CALL MOUT2 (AD,NR,NR)
150 FORMAT ('1',T50,'IDENTIFIED DAMPING MATRIX'//)
SUM=0.
WRITE (3,160)
160 FORMAT ('* MODE NUMBER*,10X,'STRUCTURAL DAMPING'//)
DO 170 I=1,NR
WRITE (3,180) I,G(I)
170 SUM=SUM+G(I)
GS=SUM/NR
180 FORMAT (18,F22.4)
WRITE (3,190) GS
190 FORMAT ('//' ,AVG STRUCTURAL DAMPING='F8.4)
200 READ (1,210) NF,IP1,IP2,NROW,NN
210 FORMAT (5I10)
IF (NF.EQ.0) GO TO 410
TORF=NKON*OT.O.AND.NROW.LE.NR
READ (1,230) (HZ(I),I=1,NF)
DO 300 L=1,NF
OMF=HZ(L)
CALL MOB2 (AM,AK,GS,NR,OMF,ZR,ZI,YR,YI,AD,NN)
IF (IP1) 220,220,280

```

```

REI 1
REI 2
REI 3
REI 4
REI 5
REI 6
REI 7
REI 8
IREI 9
ZREI 10
ZREI 11
ZREI 12
ZREI 13
ZREI 14
IREI 15
IREI 16
ZREI 17
ZREI 18
ZREI 19
ZREI 20
ZREI 21
ZREI 22
ZREI 23
ZREI 24
IREI 25
REI 26
REI 27
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REI 29
REI 30
REI 31
REI 32
REI 33
REI 34
REI 35
REI 36
REI 37
REI 38
REI 39
IREI 40
IREI 41
IREI 42
REI 43
REI 44
REI 45
REI 46
REI 47
REI 48
REI 49
REI 50
REI 51
IREI 52
IREI 53
IREI 54
IREI 55

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```

220 IF(IP2.NE.0) CALL MATAMP (HZ(L),YR,YI,NP)
    IF(IP2.NE.0) GO TO 250
    WRITE (3,240) HZ(L)
230 FORMAT (8F10.0)
240 FORMAT (1.140,'REAL MOBILITY, IMAGINARY MOBILITY   FREQ =*F10.2,
    A * HERTZ'//)
    GO TO 270
250 WRITE (3,260) HZ(L)
260 FORMAT(1.140,'ACCELERATION AMPLITUDE IN G'S, PHASE IN DEG.   FREQ
    A =*F10.2, * HERTZ'//)
270 CALL MOUT2 (YR,NR,NR)
    CALL MOUT2 (YI,NR,NR)
    GO TO 300
280 DO 290 I=1,NR
    DPR(L,I)=YR(I,I)
    DPI(L,I)=YI(I,I)
    IF(.NOT.TORF) GO TO 290
    TR(L,I)=YR(NR+I,I)
    TI(L,I)=YI(NR+I,I)
300 CONTINUE
    IF(IP1) 410,410,310
310 IF(IP2.NE.1) GO TO 330
    CALL AMP (HZ,DPR,DPI,NF,NR)
    IF(TORF) CALL AMP (HZ,TR,TI,NF,NR)
    WRITE (3,320)
320 FORMAT (1.140,'DRIVING POINT RESPONSE,  AMP IN G'S AND PHASE IN
    ADEGREES'//)
    GO TO 350
330 WRITE (3,340)
340 FORMAT (1.140,'DRIVING POINT MOBILITY,  REAL AND IMAGINARY'//)
350 CALL YOUT (HZ,DPR,NF,NR,0)
    WRITE (3,360)
360 FORMAT (1.140)
    CALL YOUT (HZ,DPI,NF,NR,IP2)
    IF(.NOT.TORF) GO TO 410
    IF (IP2.NE.1) GO TO 380
    WRITE (3,370) NPOM
370 FORMAT (1.140,'TRANSFER RESPONSE, ROW *15,*  AMP IN G'S AND PHAS
    A E IN DEG'//)
    GO TO 400
390 WRITE (3,390) NR0W
390 FORMAT (1.140,'TRANSFER MOBILITY, ROW *15,*  REAL AND IMAG'//)
400 CALL YOUT (HZ,TR,NF,NR,0)
    WRITE (3,360)
    CALL YOUT (HZ,TI,NF,NR, IP2)
410 RETURN
    END

```

```

IREI 56
IREI 57
IREI 58
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IREI 97
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IREI 99
IREI 100
IREI 101
IREI 102

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LIST OF FORTRAN SUBROUTINES

AMP Converts mobility from velocity units to acceleration as amplitude (in g's) and phase angle (in degrees)

CINV Complex inverse of complex matrix

ERR Incorporates measurement errors into simulated measurements

GEN Generalized function of form $f^T A f$ where f is a vector and A is a square matrix

INVRS Inverse of a matrix

ITER Matrix iteration for eigenvalues and eigenvectors

MITER More general iteration on product of two matrices; used for gamma iteration

MMPX Matrix multiplication

MØB Calculates complex impedance and mobility

MØUT Special output for square matrix

RANDU Random number generator

RED Removes rows and columns from matrix

YØUT Special matrix output

SYM Forms symmetric matrix from lower triangle

MØUT2 Special output for nonsquare matrix

MMPY Matrix multiplication

SITER Matrix iteration for eigenvalues and eigenvectors

MATAMP Converts velocity mobility to amplitude (g's) and phase (degrees)

YRNØM Performs normalization of mobility matrix on absolute value of largest element of mobility matrix

YRRMS Performs normalization of mobility matrix on root mean square value of mobility matrix

MØB2 Calculates complex impedance and mobility

YRFREQ Multiplies each velocity mobility matrix by its respective frequency to give acceleration mobility

REIDN Identification of mass stiffness and damping matrices

SAMPLE OUTPUT

INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT
 INXACT 9 POINT MODEL 20 POINT STRUCTURE 12/11/70

TAPE READING

EXACT DATA SIMULATED TEST
 EXACT 20 POINT UNZ 8/19/70
 20 DEGREES OF FREEDOM
 FREQUENCIES (Hz) ON TAPE

| | | | | | | | | | |
|---------|---------|---------|---------|---------|---------|---------|---------|---------|---------|
| 3.06 | 3.40 | 9.63 | 10.00 | 22.32 | 23.00 | 37.40 | 39.00 | 76.59 | 78.00 |
| 110.52 | 112.00 | 152.90 | 156.00 | 242.00 | 245.80 | 336.28 | 344.00 | 453.00 | 462.34 |
| 577.00 | 480.00 | 612.00 | 615.00 | 798.00 | 801.00 | 992.00 | 995.00 | 1226.00 | 1230.00 |
| 1456.00 | 1460.00 | 1779.00 | 1783.00 | 2442.00 | 2447.00 | 3561.00 | 3565.00 | 5440.00 | 5445.00 |

9 PJNTS TESTED
 MAX RAND ERRJK = 0.050. BIAS ERROR = 0.050 OF ELEMENTS, MAX RAND PHASE ERROR = 1.00 DEG. SEED = 206

STATIONS USED 1 3 5 8 11 13 15 18 20
 FREQUENCIES USED

| | | | | | | | | | |
|---------|---------|---------|---------|---------|---------|---------|---------|---------|---------|
| 3.06 | 3.40 | 9.63 | 10.00 | 22.32 | 23.00 | 37.40 | 39.00 | 76.59 | 78.00 |
| 110.52 | 112.00 | 152.90 | 156.00 | 242.00 | 245.80 | 336.28 | 344.00 | 453.00 | 462.34 |
| 577.00 | 480.00 | 612.00 | 615.00 | 798.00 | 801.00 | 992.00 | 995.00 | 1226.00 | 1230.00 |
| 1456.00 | 1460.00 | 1779.00 | 1783.00 | 2442.00 | 2447.00 | 3561.00 | 3565.00 | 5440.00 | 5445.00 |

IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE

9 PT MODEL 20 PT STRUCTURE 11/13/70

TAPE READING

INEXACT SIMULATED TEST DATA
 INACT 20 POINT UM2 8/19/70
 INACT 9 POINT MODEL 20 POINT STRUCTURE 17/11/70
 ORDER OF MATRICES = 9
 FREQUENCIES ON TAPE
 3.40 9.63 10.00 22.32 23.70 37.40 39.00
 110.52 152.90 156.00 242.00 245.80 336.28 344.00 78.00

FIRST PASS FREQUENCIES

3.06 3.40 9.63 10.00 22.32 23.00 37.40 39.00 76.55 76.00
 110.52 112.00 152.90 156.00 242.00 245.80 336.28 344.00 39.00 344.00

SUMMATION OF ACCELERATION MOBILITIES

ACCELERATION MOBILITY FREQ= 5.060HZ

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 1.193E-01 | 1.1885E-01 | 8.9663E-02 | 6.9142E-02 | 4.5834E-02 | 3.3404E-02 | 1.9091E-02 | -6.2770E-03 | -2.4512E-02 |
| 2 | 1.1885E-01 | 9.3381E-02 | 7.3176E-02 | 5.5235E-02 | 3.6437E-02 | 2.6138E-02 | 1.5512E-02 | -5.1528E-03 | -2.0037E-02 |
| 3 | 8.9663E-02 | 7.3176E-02 | 5.7054E-02 | 4.2072E-02 | 2.8929E-02 | 1.9390E-02 | 1.1578E-02 | -3.5627E-03 | -1.5062E-02 |
| 4 | 6.9142E-02 | 5.5235E-02 | 4.2072E-02 | 3.1536E-02 | 2.1548E-02 | 1.4896E-02 | 8.7571E-03 | -2.7923E-03 | -1.1725E-02 |
| 5 | 5.5834E-02 | 3.6437E-02 | 2.8929E-02 | 2.1548E-02 | 1.4401E-02 | 1.0729E-02 | 6.3877E-03 | -1.8185E-03 | -7.8399E-03 |
| 6 | 3.3404E-02 | 2.6138E-02 | 1.9390E-02 | 1.4896E-02 | 1.0729E-02 | 7.6637E-03 | 4.4155E-03 | -1.2225E-03 | -5.3266E-03 |
| 7 | 1.9091E-02 | 1.5512E-02 | 1.1578E-02 | 8.7571E-03 | 6.3877E-03 | 4.4155E-03 | 2.6473E-03 | -5.6178E-04 | -3.1240E-03 |
| 8 | -6.2770E-03 | -5.1528E-03 | -3.9627E-03 | -2.7923E-03 | -1.8185E-03 | -1.2225E-03 | -5.6178E-04 | 5.8744E-04 | 1.4308E-03 |
| 9 | -2.4512E-02 | -2.0037E-02 | -1.6062E-02 | -1.1725E-02 | -7.8399E-03 | -5.5266E-03 | -3.1240E-03 | 1.4308E-03 | 4.9317E-03 |

ACCELERATION MOBILITY FREQ = 3.400HZ

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 4.9517E-02 | 3.8899E-02 | 2.9917E-02 | 2.3722E-02 | 1.6061E-02 | 1.1031E-02 | 6.5267E-03 | -2.3836E-03 | -9.2107E-03 |
| 2 | 3.8899E-02 | 3.1853E-02 | 2.5367E-02 | 1.8923E-02 | 1.2607E-02 | 9.2471E-03 | 5.1950E-03 | -1.8770E-03 | -7.3050E-03 |
| 3 | 2.9917E-02 | 2.5367E-02 | 1.8712E-02 | 1.4385E-02 | 1.0263E-02 | 6.8905E-03 | 4.0612E-03 | -1.4133E-03 | -5.5762E-03 |
| 4 | 2.3722E-02 | 1.8923E-02 | 1.4385E-02 | 1.1277E-02 | 7.4347E-03 | 5.2159E-03 | 3.4579E-03 | -9.4974E-04 | -4.0157E-03 |
| 5 | 1.6061E-02 | 1.2607E-02 | 1.0263E-02 | 7.4347E-03 | 5.2510E-03 | 3.7728E-03 | 2.3305E-03 | -5.1095E-04 | -2.5540E-03 |
| 6 | 1.1031E-02 | 9.2471E-03 | 6.8905E-03 | 5.2159E-03 | 3.7728E-03 | 2.6990E-03 | 1.7381E-03 | -2.1509E-04 | -1.7582E-03 |
| 7 | 6.5267E-03 | 5.1950E-03 | 4.0012E-03 | 3.4579E-03 | 2.3305E-03 | 1.7381E-03 | 1.2589E-03 | 1.2282E-04 | -8.1639E-04 |
| 8 | -2.3836E-03 | -1.8770E-03 | -1.4133E-03 | -9.4974E-04 | -5.1095E-04 | -2.1509E-04 | 1.2282E-04 | 4.6168E-04 | 9.5873E-04 |
| 9 | -9.2107E-03 | -7.3050E-03 | -5.5762E-03 | -4.0157E-03 | -2.5540E-03 | -1.7582E-03 | -8.1639E-04 | 9.5873E-04 | 2.2743E-03 |

ACCELERATION MOBILITY FREQ= 9.630HZ

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 5.3285E-01 | 2.2601E-01 | -2.5050E-02 | -2.1061E-01 | -3.6045E-01 | -4.2201E-01 | -4.7685E-01 | -4.9652E-01 | -5.1415E-01 |
| 2 | 2.4601E-01 | 9.7757E-02 | -1.0312E-02 | -9.1107E-02 | -1.5183E-01 | -1.8930E-01 | -2.0964E-01 | -2.2340E-01 | -2.2840E-01 |
| 3 | -2.5050E-02 | -1.0312E-02 | 1.2001E-03 | 9.4715E-03 | 1.6455E-02 | 2.0479E-02 | 2.2793E-02 | 2.3613E-02 | 2.4217E-02 |
| 4 | -2.1061E-01 | -9.1107E-02 | 9.4715E-03 | 8.2921E-02 | 1.4693E-01 | 1.7400E-01 | 1.9261E-01 | 2.0940E-01 | 2.1136E-01 |
| 5 | -3.6045E-01 | -1.5183E-01 | 1.6455E-02 | 1.4693E-01 | 2.4386E-01 | 2.9491E-01 | 3.3350E-01 | 3.5820E-01 | 3.4995E-01 |
| 6 | -4.2201E-01 | -1.8930E-01 | 2.0479E-02 | 1.7400E-01 | 2.9491E-01 | 3.4664E-01 | 3.8567E-01 | 4.1743E-01 | 4.2575E-01 |
| 7 | -4.7685E-01 | -2.0964E-01 | 2.2793E-02 | 1.9261E-01 | 3.3350E-01 | 3.8567E-01 | 4.5184E-01 | 4.6989E-01 | 4.8295E-01 |
| 8 | -4.9652E-01 | -2.2340E-01 | 2.3613E-02 | 2.0940E-01 | 3.5820E-01 | 4.1743E-01 | 4.6989E-01 | 5.0327E-01 | 5.3413E-01 |
| 9 | -5.1415E-01 | -2.2840E-01 | 2.4217E-02 | 2.1136E-01 | 3.4955E-01 | 4.2575E-01 | 4.8295E-01 | 5.3413E-01 | 5.1896E-01 |

ACCELERATION MOBILITY FREQ= 10.000MHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 1.2720E-01 | 5.6919E-02 | -5.8930E-03 | -5.1901E-02 | -8.4441E-02 | -1.0464E-01 | -1.1553E-01 | -1.2500E-01 | -1.2500E-01 |
| 2 | 5.6919E-02 | 2.3756E-02 | -2.6440E-03 | -2.7296E-02 | -3.9799E-02 | -4.6932E-02 | -5.1327E-02 | -5.3489E-02 | -5.3489E-02 |
| 3 | -5.6938E-03 | -2.6440E-03 | 2.4358E-04 | 2.4210E-03 | 4.3186E-03 | 4.8332E-03 | 5.3029E-03 | 5.5197E-03 | 5.7280E-03 |
| 4 | -5.1901E-02 | -2.2296E-02 | 2.4210E-03 | 2.1823E-02 | 3.6137E-02 | 4.2779E-02 | 4.8703E-02 | 5.1147E-02 | 5.1756E-02 |
| 5 | -8.4441E-02 | -3.9799E-02 | 4.3186E-03 | 3.6137E-02 | 5.9799E-02 | 6.8708E-02 | 8.3266E-02 | 9.8020E-02 | 9.8020E-02 |
| 6 | -1.0464E-01 | -4.6932E-02 | 4.8332E-03 | 4.2779E-02 | 5.8708E-02 | 8.5224E-02 | 9.7038E-02 | 1.0570E-01 | 1.0709E-01 |
| 7 | -1.1553E-01 | -5.1327E-02 | 5.3029E-03 | 4.8703E-02 | 8.3266E-02 | 9.7038E-02 | 1.1205E-01 | 1.1335E-01 | 1.1389E-01 |
| 8 | -1.2500E-01 | -5.3489E-02 | 5.5197E-03 | 5.1147E-02 | 8.8020E-02 | 1.0570E-01 | 1.1335E-01 | 1.2078E-01 | 1.2732E-01 |
| 9 | -1.2500E-01 | -5.3489E-02 | 5.7280E-03 | 5.1756E-02 | 9.0634E-02 | 1.0709E-01 | 1.1389E-01 | 1.2732E-01 | 1.3403E-01 |

ACCELERATION MOBILITY FREQ= 22.320HZ

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 9.0943E-01 | 2.0592E-01 | -1.4744E-01 | -1.3395E-01 | 5.0079E-02 | 2.4630E-01 | 5.1907E-01 | 1.3771E 00 | 2.2491E 00 |
| 2 | 2.0542E-01 | 4.7550E-02 | -3.2826E-02 | -2.9808E-02 | 1.0760E-02 | 5.7770E-02 | 1.1476E-01 | 2.5688E-01 | 5.4112E-01 |
| 3 | -1.4744E-01 | -3.2826E-02 | 2.3849E-02 | 2.1832E-02 | -8.4392E-03 | -4.0427E-02 | -8.5937E-02 | -2.2158E-01 | -3.7340E-01 |
| 4 | -1.3395E-01 | -2.9808E-02 | 2.1832E-02 | 2.0406E-02 | -6.8461E-03 | -3.8214E-02 | -7.6723E-02 | -2.0406E-01 | -3.3329E-01 |
| 5 | 5.0079E-02 | 1.0760E-02 | -8.4392E-03 | -6.8461E-03 | 3.3310E-03 | 1.4339E-02 | 2.9377E-02 | 7.5997E-02 | 1.2343E-01 |
| 6 | 2.4630E-01 | 5.7770E-02 | -4.0427E-02 | -3.8214E-02 | 1.4339E-02 | 7.3883E-02 | 1.4762E-01 | 3.7457E-01 | 6.2337E-01 |
| 7 | 5.1907E-01 | 1.1476E-01 | -8.5937E-02 | -7.6723E-02 | 2.9377E-02 | 1.4762E-01 | 3.0351E-01 | 7.5935E-01 | 1.3658E 00 |
| 8 | 1.3771E 00 | 2.5688E-01 | -2.2158E-01 | -2.0406E-01 | 7.2997E-02 | 3.7457E-01 | 7.5935E-01 | 2.1286E 00 | 3.6141E 00 |
| 9 | 2.2491E 00 | 5.4112E-01 | -3.7340E-01 | -3.3329E-01 | 1.2343E-01 | 6.2337E-01 | 1.3658E 00 | 3.6141E 00 | 6.1281E 00 |

ACCELERATION MOBILITY FREQUENCY = 23.000HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 6.2170E-01 | 1.4324E-01 | -1.0557E-01 | -9.2943E-02 | 3.4471E-02 | 1.6881E-01 | 3.6934E-01 | 9.1123E-01 | 1.5577E 00 |
| 1.4326E-01 | 3.1780E-02 | -2.2937E-02 | -2.0792E-02 | 7.4432E-03 | 3.7747E-02 | 8.2834E-02 | 2.0373E-01 | 3.5535E-01 |
| -1.0557E-01 | -2.2937E-02 | 1.7353E-02 | 1.4192E-02 | -5.7474E-03 | -2.7116E-02 | -5.9064E-02 | -1.5041E-01 | -2.5157E-01 |
| -9.2943E-02 | -2.0792E-02 | 1.4192E-02 | 1.3626E-02 | -6.8172E-03 | -2.4314E-02 | -5.4437E-02 | -1.4155E-01 | -2.3694E-01 |
| 3.4671E-02 | 7.4432E-03 | -5.7474E-03 | -4.8172E-03 | 2.5880E-03 | 1.0261E-02 | 2.1118E-02 | 5.0205E-02 | 8.5107E-02 |
| 1.6881E-01 | 3.7747E-02 | -2.7116E-02 | -2.4314E-02 | 1.0261E-02 | 5.0301E-02 | 1.0011E-01 | 2.6034E-01 | 4.2224E-01 |
| 3.6934E-01 | 8.2834E-02 | -5.9064E-02 | -5.4437E-02 | 2.1118E-02 | 1.0011E-01 | 2.0759E-01 | 5.2815E-01 | 9.0011E-01 |
| 9.1123E-01 | 2.0373E-01 | -1.5041E-01 | -1.4155E-01 | 5.0205E-02 | 2.6034E-01 | 5.2815E-01 | 1.4774E 00 | 2.4234E 00 |
| 1.5577E 00 | 3.5535E-01 | -2.5157E-01 | -2.3694E-01 | 8.5107E-02 | 4.2224E-01 | 9.0011E-01 | 2.4934E 00 | 4.1646E 00 |

ACCELERATION MOBILITY FREQ= 37.4JOMZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 3.209E 00 | 1.1104E-01 | -6.1102E-01 | 1.3757E-01 | 1.1216E 00 | 9.6725E-01 | -5.8325E-01 | -2.325E 00 |
| 2 | 1.140E-01 | 4.1582E-03 | -2.2889E-02 | 4.8512E-03 | 3.2556E-02 | 3.8694E-02 | -2.0334E-02 | -1.0730E-01 |
| 3 | -6.1104E-01 | -2.2889E-02 | 1.2907E-01 | -2.3536E-02 | -1.3340E-01 | -2.2588E-01 | 1.1914E-01 | 6.0089E-01 |
| 4 | 1.3757E-01 | 4.8512E-03 | -2.8536E-02 | 6.6101E-03 | 4.0766E-02 | 4.9245E-02 | -2.7312E-02 | -1.2901E-01 |
| 5 | 8.5721E-01 | 3.2556E-02 | -1.8340E-01 | 4.0766E-02 | 2.6412E-01 | 3.1705E-01 | -1.7174E-01 | 8.4540E-01 |
| 6 | 1.1216E 00 | 4.0710E-02 | -2.2588E-01 | 3.1705E-01 | 4.1285E-01 | 3.5644E-01 | -2.0991E-01 | -1.0908E 00 |
| 7 | 9.6735E-01 | 3.8694E-02 | -2.0615E-01 | 2.8015E-01 | 3.5644E-01 | 3.2823E-01 | -1.8334E-01 | -9.8655E-01 |
| 8 | -5.8325E-01 | -2.0334E-02 | 1.1914E-01 | -1.7174E-01 | -2.0991E-01 | -1.8334E-01 | 1.1840E-01 | 5.8552E-01 |
| 9 | -2.325E 00 | -1.0730E-01 | 6.0089E-01 | -1.2901E-01 | -1.0908E 00 | -9.8655E-01 | 5.8552E-01 | 2.8134E 00 |

ACCELERATION VIBILITY FREQ= 39.000HZ

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 9.4367E-01 | 3.0306E-02 | -1.7166E-01 | 3.8472E-02 | 2.4890E-01 | 2.9813E-01 | 2.7501E-01 | -1.5701E-01 | -6.1298E-01 |
| 2 | 3.0306E-02 | 1.7034E-03 | -6.2990E-03 | 1.1618E-03 | 8.9874E-03 | 1.1377E-02 | 1.0195E-02 | -5.6455E-03 | -2.7880E-02 |
| 3 | -1.7166E-01 | -6.2990E-03 | 3.6168E-02 | -7.4747E-03 | -5.0047E-02 | -6.4370E-02 | -5.8873E-02 | 3.2891E-02 | 1.7087E-01 |
| 4 | 3.8472E-02 | 1.1618E-03 | -7.4747E-03 | 1.9404E-03 | 1.0431E-02 | 1.3824E-02 | 1.1711E-02 | -7.8808E-03 | -3.3933E-02 |
| 5 | 2.4890E-01 | 8.9874E-03 | -5.0047E-02 | 1.0431E-02 | 7.4925E-02 | 8.9583E-02 | 7.7492E-02 | -4.6953E-02 | -2.4520E-01 |
| 6 | 2.9813E-01 | 1.1377E-02 | -6.4370E-02 | 1.3824E-02 | 8.9583E-02 | 1.1277E-01 | 9.7422E-02 | -5.7365E-02 | -2.5629E-01 |
| 7 | 2.7501E-01 | 1.0195E-02 | -5.8873E-02 | 1.1711E-02 | 7.7492E-02 | 9.7422E-02 | 9.3349E-02 | -4.8909E-02 | -2.5750E-01 |
| 8 | -1.5701E-01 | -5.6455E-03 | 3.2891E-02 | -7.8808E-03 | -4.6953E-02 | -5.7365E-02 | -4.8909E-02 | 4.0966E-02 | 1.6138E-01 |
| 9 | -6.1298E-01 | -2.7880E-02 | 1.7087E-01 | -3.3933E-02 | -2.4520E-01 | -2.9629E-01 | -2.5950E-01 | 1.6138E-01 | 3.0850E-01 |

ACCELERATION MOBILITY FREQ = 76.590HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 5.0301E 00 | -6.9881E-01 | 1.3672E-01 | 6.2396E-01 | -8.4057E-01 | -2.0260E 00 | -2.8206E 00 | -1.5860E 00 | 2.4639E 00 |
| -8.9881E-01 | 1.3930E-01 | -2.4500E-02 | -1.0183E-01 | 1.3951E-01 | 3.4456E-01 | 4.8781E-01 | 2.7234E-01 | -4.7007E-01 |
| 1.3672E-01 | -2.4500E-02 | 4.3212E-03 | 1.5345E-02 | -2.2851E-02 | -5.2457E-02 | -7.2415E-02 | -3.5019E-01 | 7.8324E-02 |
| 6.2396E-01 | -1.0183E-01 | 1.5345E-02 | 7.3283E-02 | -9.5463E-02 | -2.2851E-01 | -3.4314E-01 | -1.5073E-01 | 3.3964E-01 |
| -8.4057E-01 | 1.3951E-01 | -2.2851E-02 | -9.5463E-02 | 1.3709E-01 | 3.1234E-01 | 4.6268E-01 | 2.4120E-01 | -4.4836E-01 |
| -2.0260E 00 | 3.4456E-01 | -5.2457E-02 | -2.2851E-01 | 3.1236E-01 | 7.6603E-01 | 1.1098E 00 | 6.1442E-01 | -1.1325E 00 |
| -2.8206E 00 | 4.8781E-01 | -7.2415E-02 | -3.4314E-01 | 4.6268E-01 | 1.1098E 00 | 1.6628E 00 | 8.5519E-01 | -1.5710E 00 |
| -1.5860E 00 | 2.7234E-01 | -3.9019E-02 | -1.9083E-01 | 2.6120E-01 | 6.1442E-01 | 8.9519E-01 | 5.0555E-01 | -9.0144E-01 |
| 2.4639E 00 | -4.7007E-01 | 7.8324E-02 | 3.3964E-01 | -4.4836E-01 | -1.1325E 00 | -1.5710E 00 | -9.0144E-01 | 1.6658E 00 |

ACCELERATION SUSCEPTIBILITY FREQUENCY = 78.000000 HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 4.209E 00 | -6.8372E-01 | 1.1061E-01 | 4.5610E-01 | -6.2667E-01 | -1.5165E 00 | -2.2052E 00 | -1.2112E 00 | 2.2209E 00 |
| -6.8372E-01 | 1.1669E-01 | -1.8803E-02 | -7.8450E-02 | 1.0401E-01 | 2.4990E-01 | 3.6650E-01 | 1.5545E-01 | -3.6934E-01 |
| 1.1061E-01 | -1.8803E-02 | 3.8547E-03 | 1.2335E-02 | -1.6988E-02 | -4.0954E-02 | -5.5847E-02 | -3.1256E-02 | 5.7637E-C2 |
| 4.5610E-01 | -7.8450E-02 | 1.2335E-02 | 5.7151E-02 | -7.4451E-02 | -1.8074E-01 | -2.6330E-01 | -1.4619E-01 | 2.5737E-01 |
| -6.2667E-01 | 1.0401E-01 | -1.6988E-02 | -7.6451E-02 | 1.0486E-01 | 2.3613E-01 | 3.4917E-01 | 1.5228E-01 | -3.5265E-01 |
| -1.5165E 00 | 2.4990E-01 | -4.0954E-02 | -1.8074E-01 | 2.3613E-01 | 5.9319E-01 | 8.7111E-01 | 4.7585E-01 | -8.9699E-01 |
| -2.2052E 00 | 3.6650E-01 | -5.5847E-02 | -2.6330E-01 | 3.4917E-01 | 8.7111E-01 | 1.2812E 00 | 7.1406E-01 | -1.2712E 00 |
| -1.2112E 00 | 1.9545E-01 | -3.1256E-02 | -1.4619E-01 | 1.9228E-01 | 4.7586E-01 | 7.1406E-01 | 4.0458E-01 | -4.8542E-01 |
| 2.2209E 00 | -3.6934E-01 | 5.7637E-02 | 2.5737E-01 | -3.5266E-01 | -8.8498E-01 | -1.2712E 00 | -6.8542E-01 | 1.2928E 00 |

ACCELERATION MOBILITY FREQ= 110.520HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 4.510E 00 | -9.6081E-01 | 6.8829E-01 | -3.6699E-01 | -1.9719E-01 | 1.0550E 00 | 2.6669E 00 | 2.9389E 00 | -3.3642E 00 |
| -9.8061E-01 | 1.9666E-01 | -1.4265E-01 | 7.8335E-02 | 4.0030E-02 | -2.2732E-01 | -5.6650E-01 | -6.6944E-01 | 6.9688E-01 |
| 6.8029E-01 | -1.4265E-01 | 9.9511E-02 | -5.6276E-02 | -2.8174E-02 | 1.6106E-01 | 3.9934E-01 | 4.5505E-01 | -4.9533E-01 |
| -3.6699E-01 | 7.8335E-02 | -5.6276E-02 | 3.1951E-02 | 1.4953E-02 | -9.0285E-02 | -2.3054E-01 | -2.4007E-01 | 2.7088E-01 |
| -1.9719E-01 | 4.0030E-02 | -2.8174E-02 | 1.4953E-02 | 1.1989E-02 | -4.4569E-02 | -1.1591E-01 | -1.3532E-01 | 1.4387E-01 |
| 1.0550E 00 | -2.2732E-01 | 1.6106E-01 | -9.0285E-02 | -4.4569E-02 | 2.6858E-01 | 6.8191E-01 | 7.1882E-01 | -8.2026E-01 |
| 2.6669E 00 | -5.6650E-01 | 3.9934E-01 | -2.3054E-01 | -1.1591E-01 | 6.8191E-01 | 1.6806E 00 | 1.8631E 00 | -2.0225E 00 |
| 2.9389E 00 | -6.6944E-01 | 4.5505E-01 | -2.4007E-01 | -1.3532E-01 | 7.1882E-01 | 1.8631E 00 | 2.0225E 00 | -2.1999E 00 |
| -3.3642E 00 | 6.9688E-01 | -4.9533E-01 | 2.7088E-01 | 1.4387E-01 | -8.2026E-01 | -2.0225E 00 | -2.1999E 00 | 2.3689E 00 |

ACCELERATION MOBILITY FREQ= 112.000HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 3.8023E 00 | -8.4398E-01 | 6.0902E-01 | -3.2547E-01 | -1.6721E-01 | 9.5518E-01 | 2.3507E 00 | 2.5649E 00 | -2.8961E 00 |
| -8.4398E-01 | 1.7383E-01 | -1.2529E-01 | 6.8903E-02 | 3.5504E-02 | -1.9648E-01 | -5.0709E-01 | -5.4303E-01 | 5.3112E-01 |
| 6.0902E-01 | -1.2529E-01 | 8.7655E-02 | -5.0907E-02 | -2.5721E-02 | 1.4537E-01 | 3.7258E-01 | 3.7607E-01 | -4.4245E-01 |
| -3.2647E-01 | 6.8903E-02 | -5.0907E-02 | 2.7235E-02 | 1.2028E-02 | -8.2950E-02 | -1.9745E-01 | -2.1340E-01 | 2.3555E-01 |
| -1.6721E-01 | 3.6506E-02 | -2.5721E-02 | 1.2828E-02 | 9.9629E-03 | -3.8399E-02 | -1.0337E-01 | -1.1922E-01 | 1.2682E-01 |
| 9.5518E-01 | -1.9648E-01 | 1.4537E-01 | -8.2950E-02 | -3.8399E-02 | 2.3070E-01 | 5.9611E-01 | 6.1406E-01 | -6.7230E-01 |
| 2.3507E 00 | -5.0709E-01 | 3.7258E-01 | -1.9745E-01 | -1.0337E-01 | 5.9611E-01 | 1.4258E 00 | 1.5358E 00 | -1.8061E 00 |
| 2.5649E 00 | -5.4303E-01 | 3.7607E-01 | -2.1340E-01 | -1.1922E-01 | 6.1406E-01 | 1.5358E 00 | 1.6782E 00 | -1.9021E 00 |
| -2.8961E 00 | 5.9312E-01 | -4.4245E-01 | 2.3595E-01 | 1.2682E-01 | -6.7230E-01 | -1.8061E 00 | -1.5021E 00 | 2.0694E 00 |

ACCELERATION MOBILITY FREQ= 152.90CHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 5 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1.4437E 00 | -3.0527E-01 | 1.5075E-01 | -2.8419E-01 | 5.9813E-01 | 5.1921E-01 | -3.3623E-01 | -1.7748E 00 | 1.5790E 00 |
| -3.0527E-01 | 6.6368E-02 | -3.1588E-02 | 6.1162E-02 | -1.2141E-01 | -1.0571E-01 | 7.1591E-02 | 3.8497E-01 | -3.4423E-01 |
| 1.5075E-01 | -3.1588E-02 | 1.4898E-02 | -3.0282E-02 | 5.9512E-01 | 5.3577E-02 | -3.2973E-02 | -1.7347E-01 | 1.6383E-01 |
| -2.8419E-01 | 6.1162E-02 | -3.0282E-02 | 5.6899E-02 | -1.2007E-01 | -1.0704E-01 | 7.1943E-02 | 3.5294E-01 | -3.2588E-01 |
| 5.9813E-01 | -1.2141E-01 | 5.9512E-01 | -1.2007E-01 | 2.4263E-01 | 2.1379E-01 | -1.4700E-01 | -7.6655E-01 | 6.5721E-01 |
| 5.1921E-01 | -1.0571E-01 | 5.3577E-02 | -1.0704E-01 | 2.1379E-01 | 1.9539E-01 | -1.1694E-01 | -6.4345E-01 | 6.0804E-01 |
| -3.3623E-01 | 7.1591E-02 | -3.2973E-02 | 7.1943E-02 | -1.4700E-01 | -1.1694E-01 | 1.1246E-01 | 4.5984E-01 | -4.2411E-01 |
| -1.7748E 00 | 3.8497E-01 | -1.7347E-01 | 3.5294E-01 | -7.6655E-01 | -6.4345E-01 | 4.5984E-01 | 2.3319E 00 | -2.1749E 00 |
| 1.5790E 00 | -3.4423E-01 | 1.6383E-01 | -3.2588E-01 | 6.5721E-01 | 6.0804E-01 | -4.2411E-01 | -2.1089E 00 | 1.8778E 00 |

ACCELERATION NOISIBILITY FREQ= 156.000CHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1.7927E 00 | -3.5091E-01 | 1.6937E-01 | -3.3707E-01 | 6.8326E-01 | 5.8932E-01 | -3.9962E-01 | -2.0208E 00 | 1.7927E 00 |
| -3.5091E-01 | 7.0943E-02 | -3.5169E-02 | 5.9292E-02 | -1.4327E-01 | -1.2652E-01 | 8.3085E-02 | 4.1769E-01 | -3.9087E-01 |
| 1.6937E-01 | -3.5169E-02 | 1.8707E-02 | -3.2317E-02 | 6.8260E-02 | 6.2447E-02 | -3.8364E-02 | -2.0134E-01 | 1.8896E-01 |
| -3.3707E-01 | 6.9292E-02 | -3.2317E-02 | 6.3047E-02 | -1.3711E-01 | -1.1935E-01 | 8.1059E-02 | 4.1256E-01 | -3.6202E-01 |
| 6.8326E-01 | -1.4327E-01 | 6.8260E-02 | -1.3711E-01 | 2.9087E-01 | 2.3847E-01 | -1.7219E-01 | -8.5338E-01 | 7.8982E-01 |
| 5.8932E-01 | -1.2652E-01 | 6.2447E-02 | -1.1935E-01 | 2.3847E-01 | 2.3039E-01 | -1.3976E-01 | -7.7002E-01 | 6.8901E-01 |
| -3.9962E-01 | 8.3085E-02 | -3.8364E-02 | 8.1059E-02 | -1.7219E-01 | -1.3976E-01 | 1.2492E-01 | 5.2734E-01 | -4.4563E-01 |
| -2.0208E 00 | 4.1769E-01 | -2.0134E-01 | 4.1256E-01 | -8.5338E-01 | -7.7002E-01 | 5.2734E-01 | 2.5955E 00 | -2.3354E 00 |
| 1.7927E 00 | -3.9087E-01 | 1.8896E-01 | -3.6202E-01 | 7.8982E-01 | 6.8901E-01 | -4.8563E-01 | -2.3354E 00 | 2.1613E 00 |

ACCELERATION MOBILITY FREQ= 242.070MHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 3.2062E-01 | -5.7173E-02 | -1.6759E-01 | 1.4652E-01 | -1.5034E-01 | -1.0438E-01 | -1.1317E-00 | 1.2022E-00 | -1.3744E-00 |
| -5.7173E-02 | 1.4048E-02 | 2.3114E-02 | -2.0100E-02 | 1.9643E-02 | 1.4197E-01 | 1.5839E-01 | -1.6633E-01 | 1.4595E-01 |
| -1.6759E-01 | 2.3114E-02 | 4.7196E-02 | -3.9824E-02 | 4.1048E-02 | 2.6814E-01 | 3.1307E-01 | -3.1920E-01 | 2.9409E-01 |
| 1.4048E-02 | -2.0100E-02 | -3.9824E-02 | 3.4260E-02 | -3.4849E-02 | -2.3209E-01 | -2.6389E-01 | 2.7033E-01 | -2.4421E-01 |
| -1.5034E-01 | 1.9643E-02 | 4.1048E-02 | -3.4849E-02 | 3.6079E-02 | 2.2676E-01 | 2.4294E-01 | -2.7506E-01 | 2.5012E-01 |
| -1.0438E-01 | 1.4197E-01 | 2.6814E-01 | -2.3209E-01 | 2.2676E-01 | 1.6179E-00 | 1.7889E-00 | -1.5441E-00 | 1.7120E-00 |
| -1.1317E-00 | 1.5839E-01 | 3.1307E-01 | -2.6389E-01 | 2.4294E-01 | 1.7889E-00 | 2.1355E-00 | -2.2607E-00 | 1.9865E-00 |
| 1.2022E-00 | -1.6633E-01 | -3.1920E-01 | 2.7033E-01 | -2.7506E-01 | -1.5441E-00 | -2.2607E-00 | 2.3530E-00 | -2.1291E-00 |
| -1.0438E-01 | 1.4595E-01 | 2.9909E-01 | -2.4421E-01 | 2.5012E-01 | 1.7120E-00 | 1.9865E-00 | -2.1291E-00 | 1.8228E-00 |

ACCELERATION QUALITY FREQ= 245.800MHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 6.2103E-01 | -6.0954E-02 | -1.2893E-01 | 1.0655E-01 | -1.0526E-01 | -7.4786E-01 | -8.6382E-01 | 9.2350E-01 | -8.0851E-01 |
| -6.0854E-02 | 8.3969E-02 | 1.7448E-02 | -1.4507E-02 | 1.4135E-02 | 1.0602E-01 | 1.2296E-01 | -1.2431E-01 | 1.1095E-01 |
| -1.2094E-01 | 1.7448E-02 | 3.4205E-02 | -3.0219E-02 | 2.9449E-02 | 2.0764E-01 | 2.2620E-01 | -2.4572E-01 | 2.1043E-01 |
| 1.0000E-01 | -1.4607E-02 | -3.0219E-02 | 2.6249E-02 | -2.6494E-02 | -1.7506E-01 | -1.8882E-01 | 2.0107E-01 | -1.7945E-01 |
| -1.0526E-01 | 1.4135E-02 | 2.9449E-02 | -2.6494E-02 | 2.5438E-02 | 1.6954E-01 | 1.8604E-01 | -2.0139E-01 | 1.8069E-01 |
| -7.4786E-01 | 1.0655E-01 | 1.0655E-01 | -1.0655E-01 | 1.6954E-01 | 1.1959E 00 | 1.3707E 00 | -1.4702E 00 | 1.2703E 00 |
| -8.0851E-01 | 1.0655E-01 | 1.0655E-01 | -1.0655E-01 | 1.8604E-01 | 1.3707E 00 | 1.5105E 00 | -1.6423E 00 | 1.4880E 00 |
| 9.2350E-01 | -1.0655E-01 | -1.0655E-01 | 1.0655E-01 | -2.0139E-01 | -1.6702E 00 | -1.6423E 00 | 1.7145E 00 | -1.5302E 00 |
| -8.0851E-01 | 1.0655E-01 | 1.0655E-01 | -1.0655E-01 | 1.8069E-01 | 1.2703E 00 | 1.4880E 00 | -1.5302E 00 | 1.4882E 00 |

ACCELERATION MOBILITY FREQ= 336.280HZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1.2719E 00 | -2.8444E-02 | -2.8397E-01 | 2.6913E-01 | -3.9221E-01 | 2.2589E-01 | 1.2460E 00 | -5.8919E-01 | 8.3975E-01 |
| -2.8444E-02 | 1.9030E-03 | 1.5833E-02 | -1.5807E-02 | 2.2582E-02 | -1.2607E-02 | -7.4099E-02 | 3.4405E-02 | -4.9299E-02 |
| -2.8397E-01 | 1.5833E-02 | 1.2582E-01 | -1.1532E-01 | 1.7727E-01 | -9.9707E-02 | -5.4298E-01 | 2.6262E-01 | -3.6921E-01 |
| 2.6913E-01 | -1.5807E-02 | 1.1532E-01 | 1.1712E-01 | -1.7182E-01 | 9.5037E-02 | 5.3327E-01 | -2.5400E-01 | 3.7982E-01 |
| -3.9221E-01 | 2.2582E-02 | 1.7727E-01 | -1.7182E-01 | 2.5973E-01 | -1.4389E-01 | -8.1980E-01 | 3.4409E-01 | -5.8096E-01 |
| 2.2582E-01 | -1.2607E-02 | -9.9707E-02 | 9.5027E-02 | -1.4389E-01 | 9.7694E-02 | 4.6736E-01 | -2.2702E-01 | 3.0985E-01 |
| 1.2460E 00 | -7.4099E-02 | 5.4298E-01 | 5.3327E-01 | -8.1980E-01 | 4.6736E-01 | 2.5672E 00 | -1.2598E 00 | 1.7365E 00 |
| -5.8919E-01 | 3.4405E-02 | 2.6262E-01 | -2.5400E-01 | 3.9409E-01 | -2.2702E-01 | -1.2598E 00 | 6.2600E-01 | -8.3557E-01 |
| 8.3975E-01 | -4.9299E-02 | -3.6921E-01 | 3.7992E-01 | -5.8096E-01 | 3.0985E-01 | 1.7365E 00 | -8.3557E-01 | 1.2573E 00 |

ACCELERATION MOBILITY FREQ= 344.000MHZ

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 1.4848E 00 | -3.4613E-02 | -3.1697E-01 | 3.0634E-01 | -4.5154E-01 | 2.7267E-01 | 1.4385E 00 | -6.8701E-01 | 4.6742E-01 |
| 2 | -3.4613E-02 | 4.8338E-03 | 1.7505E-02 | -1.8028E-02 | 2.5828E-02 | -1.4060E-02 | -8.6340E-02 | 3.8348E-02 | -5.7468E-02 |
| 3 | -3.1697E-01 | 1.7505E-02 | 1.3867E-01 | -1.3308E-01 | 2.0320E-01 | -1.1229E-01 | -6.5877E-01 | 3.0103E-01 | -4.4151E-01 |
| 4 | 3.0634E-01 | -1.8028E-02 | 1.3308E-01 | 1.3439E-01 | -1.9488E-01 | 1.0824E-01 | 6.5069E-01 | -2.5280E-01 | 4.4419E-01 |
| 5 | -4.5154E-01 | 2.5828E-02 | 2.0320E-01 | -1.9488E-01 | 3.0409E-01 | -1.6370E-01 | -9.4750E-01 | 4.3668E-01 | -6.5066E-01 |
| 6 | 2.7267E-01 | -1.4060E-02 | 1.1229E-01 | 1.0824E-01 | -1.6370E-01 | 1.086E-01 | 5.0545E-01 | -2.5535E-01 | 3.5617E-01 |
| 7 | 1.4385E 00 | -8.6340E-02 | -6.5877E-01 | 6.5069E-01 | -9.4750E-01 | 5.0545E-01 | 3.0855E 00 | -1.4322E 00 | 2.0706E 00 |
| 8 | -6.8701E-01 | 3.8348E-02 | 3.0103E-01 | -2.9280E-01 | 4.3668E-01 | -2.5535E-01 | -1.4322E 00 | 7.2430E-01 | -9.6030E-01 |
| 9 | 4.6742E-01 | -5.7468E-02 | -4.4151E-01 | 4.4419E-01 | -6.5066E-01 | 3.5617E-01 | 2.0706E 00 | -9.6030E-01 | 1.3548E 00 |

SUM OF REAL MOBILITIES

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 3.139E 01 | -3.3294E 00 | 2.0249E-02 | 3.7338E-01 | -8.4168E-01 | -3.6624E-01 | 1.5105E 00 | 6.4850E-01 | 1.6468E 00 |
| 2 | -3.3294E 00 | 1.1229E 00 | -3.0346E-01 | -5.4966E-02 | 5.4729E-02 | 1.0650E-01 | 5.6523E-02 | 9.0789E-02 | 3.1609E-01 |
| 3 | 2.0249E-02 | -3.0346E-01 | 8.5829E-01 | -3.9219E-01 | 2.9734E-01 | 2.8688E-01 | -4.5644E-01 | 1.8857E-01 | -5.9186E-01 |
| 4 | 3.7338E-01 | -5.4966E-02 | 3.9219E-01 | 8.1177E-01 | -5.7577E-01 | -7.7532E-01 | 2.7618E-02 | -2.4436E-01 | 3.2975E-01 |
| 5 | -8.4168E-01 | 5.4729E-02 | 2.9734E-01 | -5.7577E-01 | 2.0948E 00 | 1.8158E 00 | -2.2332E-01 | -7.1829E-01 | -3.4459E-01 |
| 6 | -3.6624E-01 | 1.0650E-01 | 2.8688E-01 | -7.7532E-01 | 1.8158E 00 | 6.3986E 00 | 8.3252E 00 | -2.0027E 00 | 1.6194E 00 |
| 7 | 1.5105E 00 | 5.6523E-02 | -4.5644E-01 | 2.7618E-02 | -2.2332E-01 | 8.3252E 00 | 1.7087E 01 | 1.0783E 00 | 1.3137E 00 |
| 8 | 6.4850E-01 | 9.0789E-02 | 1.8857E-01 | -2.2436E-01 | -7.1829E-01 | -2.0027E 00 | 1.0783E 00 | 1.5403E 01 | -8.0701E 00 |
| 9 | 1.6468E 00 | 3.1609E-01 | -5.9186E-01 | 3.2975E-01 | -3.3659E-01 | 1.6194E 00 | 1.3137E 00 | -8.0701E 00 | 3.1938E 01 |

INVERSE OF SUM OF REAL MOB

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 5.0050E-04 | 1.6690E-01 | 6.0446E-02 | 3.2791E-02 | 2.7644E-03 | 1.2553E-02 | -8.9994E-03 | -2.5677E-03 | -4.3368E-03 |
| 2 | 1.8690E-01 | 1.6438E 00 | 8.3188E-01 | 2.7081E-01 | 3.1200E-01 | -3.8553E-01 | 1.9853E-01 | -6.3128E-02 | -1.3536E-02 |
| 3 | 6.0446E-02 | 8.3188E-01 | 2.1800E 00 | 4.8681E-01 | 9.2302E-01 | -1.2304E 00 | 6.0695E-01 | -1.2721E-01 | 3.4035E-02 |
| 4 | 3.2791E-02 | 2.7081E-01 | 4.8682E-01 | 2.5925E 00 | -8.6144E-01 | 1.7123E 00 | -8.5079E-01 | 2.0168E-01 | -3.2060E-02 |
| 5 | 2.7644E-03 | 3.1200E-01 | 9.2302E-01 | -8.6144E-01 | 3.4414E 00 | -3.7410E 00 | 1.9084E 00 | -2.5205E 00 | 5.0556E-01 |
| 6 | 1.2553E-02 | -3.8553E-01 | -1.2304E 00 | 1.7123E 00 | -3.7410E 00 | 4.9497E 00 | -2.5205E 00 | 5.0556E-01 | -9.6286E-02 |
| 7 | -8.9994E-03 | 1.9853E-01 | 6.0695E-01 | -8.5079E-01 | 1.9084E 00 | -2.5205E 00 | 1.3438E 00 | -2.6197E-01 | 4.6083E-02 |
| 8 | -2.5677E-03 | -1.3536E-02 | -1.3721E-01 | 2.0168E-01 | -3.5140E-01 | 5.0556E-01 | -2.6197E-01 | 1.1173E-01 | 5.8028E-03 |
| 9 | -4.3368E-03 | -1.3536E-02 | 3.4035E-02 | -3.2060E-02 | 8.1431E-02 | -9.6286E-02 | 4.6083E-02 | 5.8028E-03 | 3.7941E-02 |

SECOND PASS FREQUENCIES

| | | | | | | | | |
|---|------|-------|-------|-------|--------|--------|--------|--------|
| 1 | 9.63 | 22.32 | 37.40 | 75.59 | 110.52 | 152.90 | 242.00 | 336.28 |
|---|------|-------|-------|-------|--------|--------|--------|--------|

ITERATED PHI

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 1.0000E 00 | -9.9511E-01 | 3.7916E-01 | 1.0000E 00 | 1.0000E 00 | 1.0000E 00 | -7.6925E-01 | 5.8849E-01 | 4.9849E-01 |
| 2 | 8.0187E-01 | -4.3311E-01 | 8.6950E-02 | 3.6254E-02 | -1.6546E-01 | -2.0947E-01 | 1.6460E-01 | -8.0119E-02 | -2.8434E-02 |
| 3 | 6.2394E-01 | 4.5990E-02 | -6.1828E-02 | -2.0193E-01 | 2.6481E-02 | 1.4995E-01 | -7.7471E-02 | -1.3341E-01 | -2.1495E-01 |
| 4 | 4.6676E-01 | 4.0241E-01 | -5.6207E-02 | 4.4657E-02 | 1.2476E-01 | -8.2675E-02 | 1.5151E-01 | 1.2067E-01 | 2.0989E-01 |
| 5 | 3.1594E-01 | 6.8590E-01 | 2.0501E-02 | 2.8493E-01 | -1.6857E-01 | -4.3212E-02 | -3.1909E-01 | -1.2413E-01 | -3.2167E-01 |
| 6 | 2.2012E-01 | 8.2276E-01 | 1.0470E-01 | 3.6712E-01 | -3.9323E-01 | 2.4179E-01 | -2.7802E-01 | -8.8929E-01 | 1.7800E-01 |
| 7 | 1.3013E-01 | 9.1414E-01 | 2.2319E-01 | 3.2192E-01 | -5.8562E-01 | 6.0595E-01 | 1.9310E-01 | -6.5086E-01 | 1.0000E 00 |
| 8 | -4.3010E-02 | 9.8684E-01 | 5.8933E-01 | -1.9167E-01 | -3.2138E-01 | 6.6418E-01 | 1.0000E 00 | 1.0000E 00 | -4.8853E-01 |
| 9 | -1.7275E-01 | 1.0000E 00 | 1.0000E 00 | -9.4863E-01 | 5.4342E-01 | -7.3634E-01 | -8.5744E-01 | -8.7196E-01 | 6.8380E-01 |

ITERATIONS

| | 4 | 5 | 7 | 9 | 8 | 6 | 19 | 5 |
|----|---|---|---|---|---|---|----|---|
| 4 | | | | | | | | |
| 5 | | | | | | | | |
| 7 | | | | | | | | |
| 9 | | | | | | | | |
| 8 | | | | | | | | |
| 6 | | | | | | | | |
| 19 | | | | | | | | |
| 5 | | | | | | | | |

ITERATED GAMMA

| | | | | | | | | | |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 5.2440E-02 | -7.2297E-02 | 2.2378E-01 | 2.3126E-01 | 2.5506E-01 | 1.7703E-01 | -9.9581E-02 | 3.6486E-02 | 2.8518E-02 |
| 2 | 4.8192E-01 | -9.8914E-01 | 1.3357E 00 | 4.7087E-01 | -5.9484E-01 | -8.1112E-01 | 7.1786E-01 | -1.3483E-01 | -2.1428E-01 |
| 3 | 5.3050E-01 | 3.1730E-03 | -7.8115E-01 | -1.3570E 00 | 3.7819E-01 | 1.0712E 00 | 3.3268E-02 | -1.6758E-01 | -7.2743E-01 |
| 4 | 3.0927E-01 | 7.4488E-01 | -8.4920E-01 | 7.1016E-01 | 2.0865E 00 | -6.2510E-01 | 1.5099E 00 | -8.8758E-01 | 3.3187E-01 |
| 5 | 2.0680E-01 | 2.1202E-01 | -1.4160E-02 | 3.9665E-03 | 2.0016E 00 | -3.7478E-01 | -1.6714E 00 | 1.5814E 00 | -1.4664E-01 |
| 6 | 1.5139E-01 | 2.6002E-01 | 3.3167E-01 | 1.1359E 00 | 2.0193E 00 | 4.3968E-02 | 1.2724E 00 | -2.6089E 00 | -6.5871E-01 |
| 7 | 8.0815E-02 | -9.1228E-02 | -1.3295E-01 | -5.1532E-01 | -1.3776E 00 | 2.9727E-01 | -5.7806E-01 | 1.1001E 00 | 6.6482E-01 |
| 8 | -1.0527E-02 | 1.0366E-01 | 5.1809E-01 | 2.6782E-02 | 1.0607E-01 | 1.9122E-01 | 3.8372E-01 | -7.4285E-02 | -2.2756E-01 |
| 9 | -1.1870E-03 | 9.0643E-02 | 4.1899E-01 | -2.0958E-01 | 1.0644E-01 | -1.4503E-01 | -1.4859E-01 | -1.0119E-02 | 6.1432E-02 |

ITERATIONS

| | | | | | | | | |
|---|---|---|----|---|---|---|----|---|
| 4 | 5 | 6 | 12 | 7 | 6 | 7 | 18 | 7 |
|---|---|---|----|---|---|---|----|---|

GAMMA = PHI INVERSE TRANSPOSE

| | | | | | | | | | |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 4.9771E-02 | -7.6608E-02 | 2.4300E-01 | 2.3003E-01 | 2.5964E-01 | 1.9220E-01 | -9.0876E-02 | 3.4732E-02 | 1.3334E-02 |
| 2 | 4.6532E-01 | -4.7800E-01 | 9.6813E-01 | 3.7568E-01 | -6.4113E-01 | -7.2242E-01 | 3.3136E-01 | -8.1442E-02 | -3.4852E-02 |
| 3 | 6.0554E-01 | 2.9031E-02 | -9.7707E-01 | -1.6855E 00 | -4.5391E-01 | 1.1964E 00 | -8.6034E-01 | 1.5453E-01 | -3.1642E-02 |
| 4 | 2.7163E-01 | 7.0803E-01 | -6.3625E-01 | 9.2060E-01 | 2.3581E 00 | -6.8761E-01 | 1.8234E 00 | -7.5178E-01 | -9.6828E-02 |
| 5 | 4.3780E-01 | 3.1369E-01 | -5.4040E-01 | -8.4248E-01 | -3.4520E 00 | -2.4200E-01 | -3.1878E 00 | 2.6800E 00 | 1.1763E 00 |
| 6 | -4.4492E-01 | 1.2515E-01 | 9.8582E-01 | 2.0911E 00 | 3.8741E 00 | -1.4160E-01 | 3.2939E 00 | -3.3677E 00 | -2.2313E 00 |
| 7 | 2.4787E-01 | -2.4423E-02 | -5.0417E-01 | -1.0536E-01 | -2.3797E 00 | 3.9980E-01 | -1.7018E 00 | 1.5347E 00 | 1.5049E 00 |
| 8 | -5.6670E-02 | 9.1982E-02 | 5.9028E-01 | 1.6330E-01 | 3.7812E-01 | 1.5409E-01 | 5.9319E-01 | -1.8539E-01 | -4.1266E-01 |
| 9 | 9.0549E-03 | 4.3910E-02 | 4.0002E-01 | -2.3290E-01 | 3.5203E-02 | -1.3499E-01 | -1.9435E-01 | 2.3353E-02 | 1.0512E-01 |

THIRD PASS FREQUENCIES

| | | | | | | | |
|--------|--------|--------|--------|--------|--------|-------|--------|
| 5.06 | 3.40 | 9.63 | 10.00 | 22.32 | 37.40 | 76.55 | 78.00 |
| 110.52 | 112.00 | 152.90 | 156.00 | 242.00 | 336.28 | 39.00 | 344.00 |

YSTAR USING ITERATED GAMMA

| MODE | UM 1 | JM 2 | REAL (OM 1) | YSTAR (MODE) (OM 2) | IMAG (OM 1) | (OM 2) | REAL (OM 1) | ZSTAR (MODE) (JM 2) | IMAG (OM 1) | (OM 2) |
|------|--------|--------|-------------|------------------------|-------------|-------------|-------------|------------------------|-------------|------------|
| 1 | 3.00 | | 4.7737E-02 | 1.4685E-02 | 6.0094E-02 | -4.0989E-02 | 8.1046E 00 | 7.7461E 00 | -1.0202E 01 | |
| 2 | 9.03 | 3.40 | 5.4232E-02 | 1.2909E-02 | -3.5918E-02 | -2.9742E-02 | 1.2817E 01 | 1.2279E 01 | 8.4888E 00 | 2.1621E 01 |
| 3 | 22.32 | 10.00 | 2.7338E-01 | 1.8165E-01 | 9.5339E-02 | -1.5671E-01 | 3.2613E 00 | 3.1561E 00 | -1.1373E 00 | 2.8293E 01 |
| 4 | 37.40 | 23.00 | 8.2268E-02 | 2.1644E-02 | 1.2287E-03 | -3.7290E-02 | 1.2153E 01 | 1.1696E 01 | -1.8151E-01 | 2.7227E 00 |
| 5 | 76.59 | 59.00 | 6.2450E-02 | 4.6673E-02 | 9.8975E-03 | -2.7702E-02 | 1.5620E 01 | 1.5844E 01 | -2.4756E 00 | 1.9966E 01 |
| 6 | 110.22 | 70.00 | 4.0707E-02 | 3.5185E-02 | 4.7320E-03 | -1.4738E-02 | 2.4238E 01 | 2.4179E 01 | -2.8176E 00 | 9.4042E 00 |
| 7 | 152.40 | 112.00 | 1.5707E-02 | 1.7655E-02 | 7.5959E-03 | -5.8243E-03 | 5.1599E 01 | 5.1082E 01 | -2.4954E 01 | 1.0123E 01 |
| 8 | 242.00 | 150.00 | 7.5210E-03 | 3.8804E-03 | 1.8826E-03 | -8.4479E-04 | 1.2511E 02 | 2.4604E 02 | -3.1333E 01 | 1.6852E 01 |
| 9 | 336.28 | 245.80 | 7.5837E-03 | 8.9333E-03 | 4.1601E-03 | -3.2302E-03 | 1.0136E 02 | 9.8997E 01 | -5.5602E 01 | 5.3564E 01 |
| | | 344.00 | | | | | | | | 3.5796E 01 |

GENERALIZED MASSES AND NATURAL FREQUENCIES

| MODE | GEN MASS | NAT FREQ |
|------|----------|-----------|
| 1 | 7.5891 | 3.16517 |
| 2 | 4.4084 | 9.47552 |
| 3 | 0.4545 | 22.51922 |
| 4 | 1.0276 | 37.41409 |
| 5 | 0.6743 | 76.89167 |
| 6 | 0.6987 | 110.84041 |
| 7 | 1.0711 | 154.74289 |
| 8 | 1.7815 | 243.19555 |
| 9 | 0.9399 | 340.95532 |

IDENTIFIED MASS MATRIX

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 9 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 2.0624E-01 | 2.8535E-01 | -1.1160E-01 | 1.0187E-01 | -3.3755E-01 | 4.7988E-01 | -2.7881E-01 | 6.1291E-02 | -1.1709E-02 |
| 2 | 2.8535E-01 | 3.9931E 00 | 2.6458E-01 | -3.6572E-01 | 3.7867E-01 | -4.6971E-01 | 2.4693E-01 | -6.1698E-02 | 3.1051E-03 |
| 3 | -1.1160E-01 | 2.6458E-01 | 8.0997E 00 | -3.1569E 00 | 8.2370E 00 | -1.1269E 01 | 5.2004E 00 | -1.3643E 00 | 3.2976E-01 |
| 4 | 1.0187E-01 | -3.6575E-01 | 3.1569E 00 | 1.2585E 01 | -1.4237E 01 | 1.8767E 01 | -1.5020E 01 | 2.1376E 00 | -4.8045E-01 |
| 5 | -3.3755E-01 | 3.7867E-01 | 8.2370E 00 | -1.4237E 01 | 3.5797E 01 | -4.2128E 01 | 2.2099E 01 | -4.6189E 00 | 1.0253E 00 |
| 6 | 4.7988E-01 | -4.6971E-01 | -1.1269E 01 | 1.8767E 01 | -4.2128E 01 | 5.3157E 01 | -2.7970E 01 | 5.8978E 00 | -1.2660E 00 |
| 7 | -2.7881E-01 | 2.4693E-01 | 6.2004E 00 | -1.0020E 01 | 2.2099E 01 | -2.7970E 01 | 1.5096E 01 | -3.1643E 00 | 6.4651E-01 |
| 9 | 6.1291E-02 | -6.1698E-02 | -1.3643E 00 | 2.1376E 00 | -4.6189E 00 | 5.8978E 00 | -3.1643E 00 | 9.5834E-01 | -9.5173E-02 |
| 9 | -1.1709E-02 | 3.1052E-03 | 3.2976E-01 | -4.8045E-01 | 1.0253E 01 | -1.2660E 00 | 6.4651E-01 | -9.5173E-02 | 2.6270E-01 |

IDENTIFIED STIFFNESS MATRIX

| | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 4.1727E 04 | -1.1068E 05 | 1.3412E 05 | -2.2876E 05 | 6.0884E 05 | -7.9678E 05 | 4.1736E 05 | -8.6880E 04 | 2.6446E 04 |
| 2 | -1.1368E 05 | 4.0408E 05 | -6.2683E 05 | 8.3343E 05 | -1.7693E 06 | 2.2743E 06 | -1.2000E 06 | 2.5555E 05 | -6.1074E 04 |
| 3 | 1.3413E 05 | -6.2682E 05 | 1.5403E 06 | -2.6108E 06 | 4.5686E 06 | -5.2657E 06 | 2.6983E 06 | -5.6418E 05 | 1.3120E 05 |
| 4 | -2.2674E 05 | 8.3331E 05 | -2.6106E 06 | 7.1041E 06 | -1.6450E 07 | 1.9660E 07 | -9.8426E 06 | 1.5858E 06 | -4.4839E 05 |
| 5 | 6.0977E 05 | -1.7689E 06 | 4.5679E 06 | -1.6450E 07 | 4.8033E 07 | -6.1636E 07 | 3.1518E 07 | -6.2949E 06 | 1.4202E 06 |
| 6 | -7.9669E 05 | 2.2738E 06 | -5.2648E 06 | 1.9659E 07 | -6.1636E 07 | 8.2183E 07 | -4.3214E 07 | 8.7834E 06 | -1.9802E 06 |
| 7 | 4.1731E 05 | -1.1998E 06 | 2.6978E 06 | -9.8422E 06 | 3.1518E 07 | -4.3214E 07 | 2.3485E 07 | -5.0115E 06 | 1.1456E 06 |
| 8 | -8.6870E 04 | 2.5550E 05 | -5.6407E 05 | 1.9857E 06 | -6.2949E 06 | 8.7835E 06 | -5.0115E 06 | 1.2678E 06 | -3.2644E 05 |
| 9 | 2.0444E 04 | -6.1061E 04 | 1.3118E 05 | -4.4837E 05 | 1.4202E 06 | -1.9802E 06 | 1.1455E 06 | -3.2644E 05 | 9.9031E 04 |

IDENTIFIED DAMPING MATRIX

| MODE NUMBER | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 1 | 2.1150E 03 | -8.8975E 03 | -2.8250E 03 | -7.5527E 03 | 1.6145E 04 | -2.3640E 04 | 1.2446E 04 | -2.5518E 03 | 7.1254E 02 |
| 2 | -8.8975E 03 | 5.3912E 04 | 2.0394E 04 | 5.4405E 04 | -8.9268E 04 | 1.3436E 04 | -7.7911E 04 | 2.3160E 04 | -6.4418E 03 |
| 3 | -2.8250E 03 | 2.0394E 04 | 1.4470E 05 | -3.9253E 04 | -5.520E 04 | 1.8988E 05 | -1.3610E 05 | 4.1913E 04 | -1.1143E 04 |
| 4 | -7.5527E 03 | 5.4405E 04 | -3.9253E 04 | 2.7207E 05 | -4.2286E 05 | 4.1935E 05 | -1.6184E 05 | 2.0511E 04 | -2.4453E 03 |
| 5 | 1.6145E 04 | -8.9568E 04 | -5.5220E 04 | -4.2286E 05 | 9.2250E 05 | -1.0986E 06 | 4.6051E 05 | -5.5435E 04 | 5.8816E 03 |
| 6 | -2.3640E 04 | 1.3436E 04 | 1.8988E 05 | 4.1935E 05 | -1.0986E 06 | 1.5021E 06 | -6.5985E 05 | 9.5109E 04 | -1.2117E 04 |
| 7 | 1.2446E 04 | -7.7911E 04 | -1.3610E 05 | 1.6184E 05 | 4.6051E 05 | -6.9855E 05 | 3.5796E 05 | -5.5163E 04 | 9.3391E 03 |
| 8 | -2.5518E 03 | 2.3160E 04 | 4.1913E 04 | 2.0511E 04 | -5.5435E 04 | 9.5109E 04 | -5.9163E 04 | 2.0273E 04 | -5.9727E 03 |
| 9 | 7.1254E 02 | -6.4418E 03 | -1.1143E 04 | -2.4453E 03 | 5.8816E 03 | -1.2117E 04 | 9.3391E 03 | -5.9727E 03 | 2.5683E 03 |

STRUCTURAL DAMPING

| | |
|---|--------|
| 1 | 0.0519 |
| 2 | 0.0496 |
| 3 | 0.0503 |
| 4 | 0.0505 |
| 5 | 0.0478 |
| 6 | 0.0497 |
| 7 | 0.0490 |
| 8 | 0.0457 |
| 9 | 0.0497 |

AVG STRUCTURAL DAMPING= 0.0493

DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN DEGREES

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-------|------------|------------|------------|------------|------------|------------|------------|------------|------------|
| 3.000 | 2.4025E-03 | 1.7516E-03 | 1.0540E-03 | 5.9907E-04 | 2.9621E-04 | 1.7662E-04 | 1.0808E-04 | 1.0808E-04 | 1.0808E-04 |
| 3.100 | 5.2659E-03 | 3.3468E-03 | 2.0232E-03 | 1.1407E-03 | 5.3777E-04 | 2.9136E-04 | 1.4564E-04 | 1.1686E-04 | 1.1686E-04 |
| 3.200 | 6.4026E-03 | 4.1386E-03 | 2.5072E-03 | 1.4012E-03 | 6.2309E-04 | 2.9172E-04 | 1.0800E-04 | 1.1451E-04 | 1.1451E-04 |
| 3.300 | 7.1206E-03 | 4.5819E-03 | 3.5120E-03 | 1.7673E-03 | 6.9095E-04 | 5.0846E-05 | 1.0899E-04 | 1.5244E-04 | 1.5244E-04 |
| 3.400 | 8.9205E-03 | 5.2788E-03 | 4.7849E-03 | 2.2788E-03 | 8.5859E-04 | 4.3145E-04 | 3.5774E-04 | 5.4075E-04 | 5.4075E-04 |
| 3.500 | 1.3054E-03 | 4.8285E-05 | 1.5747E-04 | 1.4803E-04 | 6.3975E-04 | 9.7385E-04 | 1.2516E-03 | 1.6868E-03 | 2.3594E-03 |
| 3.600 | 4.6616E-03 | 6.7221E-04 | 1.3125E-04 | 6.9225E-04 | 2.1912E-03 | 3.2039E-03 | 4.0119E-03 | 4.9526E-03 | 5.8686E-03 |
| 3.700 | 1.1323E-02 | 2.0998E-03 | 1.3349E-04 | 1.9225E-03 | 5.3654E-03 | 7.7334E-03 | 9.5773E-03 | 1.1259E-02 | 1.1909E-02 |
| 3.800 | 1.6633E-02 | 2.2676E-03 | 1.6274E-04 | 1.9246E-03 | 5.5348E-03 | 7.9636E-03 | 9.8776E-03 | 1.1428E-02 | 1.1700E-02 |
| 3.900 | 4.9459E-03 | 1.1736E-03 | 1.4299E-04 | 8.9329E-04 | 3.4152E-03 | 4.1512E-03 | 4.1512E-03 | 4.5006E-03 | 3.7243E-03 |
| 4.000 | 1.1490E-03 | 5.0421E-04 | 1.2416E-04 | 3.2089E-04 | 7.3705E-04 | 9.8373E-04 | 1.1070E-03 | 6.9013E-04 | 1.0623E-03 |
| 4.100 | 3.2545E-04 | 1.0998E-04 | 1.0998E-04 | 2.3837E-04 | 5.0351E-04 | 6.2700E-04 | 3.0346E-04 | 3.0346E-04 | 2.9557E-03 |
| 4.200 | 4.4265E-04 | 3.4419E-04 | 9.2664E-05 | 1.9994E-04 | 4.0319E-04 | 4.5169E-04 | 3.2072E-04 | 1.2442E-03 | 5.4997E-03 |
| 4.300 | 1.4027E-03 | 2.9109E-04 | 6.7174E-05 | 1.6858E-04 | 3.3997E-04 | 3.0831E-04 | 9.3041E-05 | 2.8255E-03 | 1.0009E-02 |
| 4.400 | 3.3699E-03 | 2.0192E-04 | 2.6386E-05 | 1.2730E-04 | 2.8703E-04 | 1.3197E-04 | 6.8183E-04 | 6.6528E-03 | 2.1062E-02 |
| 4.500 | 1.2279E-02 | 4.5845E-04 | 2.6649E-04 | 1.8998E-04 | 2.2899E-04 | 7.6085E-04 | 3.8706E-03 | 2.7790E-02 | 8.1487E-02 |
| 4.600 | 4.4412E-02 | 8.4462E-04 | 4.2282E-04 | 3.6339E-04 | 2.3720E-04 | 1.2217E-03 | 5.5453E-03 | 3.8726E-02 | 1.1200E-01 |
| 4.700 | 9.2527E-02 | 9.2527E-04 | 4.4852E-04 | 4.0216E-04 | 2.4346E-04 | 1.2997E-03 | 5.7628E-03 | 4.0003E-02 | 1.1530E-01 |
| 4.800 | 1.2460E-02 | 9.4042E-04 | 3.9862E-04 | 4.2802E-04 | 2.5339E-04 | 1.1355E-03 | 4.6794E-03 | 3.1747E-02 | 9.0001E-02 |
| 4.900 | 1.3638E-03 | 4.9015E-04 | 1.2129E-04 | 2.3939E-04 | 1.5005E-04 | 3.0501E-04 | 1.1771E-03 | 8.1654E-03 | 2.0842E-02 |
| 5.000 | 2.7793E-03 | 4.0122E-04 | 2.9547E-05 | 1.9679E-04 | 7.2951E-05 | 1.8298E-04 | 3.7205E-04 | 4.6717E-03 | 9.2729E-03 |
| 5.100 | 7.5268E-03 | 3.6683E-04 | 1.9622E-04 | 1.7539E-04 | 4.1703E-04 | 8.0507E-04 | 3.7773E-04 | 3.6203E-03 | 3.8052E-03 |
| 5.200 | 2.5832E-02 | 3.2818E-04 | 9.6681E-04 | 1.4180E-04 | 1.9071E-03 | 3.2967E-03 | 2.4298E-03 | 2.6601E-03 | 1.7009E-02 |
| 5.300 | 4.5695E-02 | 3.3047E-04 | 1.8106E-03 | 1.6560E-04 | 3.6159E-03 | 6.0741E-03 | 4.6635E-03 | 3.0421E-03 | 3.8698E-02 |
| 5.400 | 5.1495E-02 | 3.5340E-04 | 2.0948E-03 | 2.0481E-04 | 4.1659E-03 | 6.9251E-03 | 5.3682E-03 | 3.7356E-03 | 4.7072E-02 |
| 5.500 | 4.4894E-02 | 3.7860E-04 | 1.9056E-03 | 2.1138E-03 | 3.7772E-03 | 6.1741E-03 | 5.4036E-03 | 3.8983E-03 | 4.8069E-02 |
| 5.600 | 1.9231E-02 | 3.5438E-04 | 9.2353E-04 | 2.0636E-04 | 1.8087E-03 | 2.8044E-03 | 2.1668E-03 | 3.5666E-03 | 2.5030E-02 |
| 5.700 | 7.3461E-03 | 3.0825E-04 | 4.7324E-04 | 1.7041E-04 | 9.0052E-04 | 1.2070E-03 | 7.1059E-04 | 2.7243E-03 | 1.4333E-02 |
| 5.800 | 3.8222E-03 | 2.6644E-04 | 3.6708E-04 | 1.4737E-04 | 6.7252E-04 | 7.2404E-04 | 1.5703E-04 | 2.2239E-03 | 1.1308E-02 |
| 5.900 | 1.9723E-03 | 1.4263E-04 | 2.8176E-04 | 8.5181E-05 | 4.3294E-04 | 1.5591E-04 | 1.5244E-03 | 1.2306E-03 | 7.9836E-03 |
| 6.000 | 1.5595E-02 | 2.7912E-04 | 2.2468E-04 | 1.3350E-04 | 1.5579E-04 | 2.1400E-03 | 6.1146E-03 | 9.7195E-04 | 3.5148E-03 |
| 6.100 | 6.8763E-02 | 1.9140E-03 | 1.9424E-03 | 1.0227E-03 | 1.8219E-03 | 1.0475E-02 | 2.3906E-02 | 7.1277E-03 | 1.8795E-03 |
| 6.200 | 7.7944E-02 | 2.2448E-03 | 2.1056E-03 | 1.2136E-03 | 2.2476E-03 | 1.2024E-02 | 2.6763E-02 | 8.2220E-03 | 2.3998E-02 |
| 6.300 | 7.8299E-02 | 2.2630E-03 | 2.1291E-04 | 1.2270E-03 | 2.2760E-03 | 1.2082E-02 | 2.6824E-02 | 8.2588E-03 | 2.4421E-02 |
| 6.400 | 7.8418E-02 | 2.2770E-03 | 2.1519E-04 | 1.2338E-03 | 2.3070E-03 | 1.2133E-02 | 2.6837E-02 | 8.2960E-03 | 2.4797E-02 |
| 6.500 | 4.3808E-02 | 1.4087E-03 | 2.1239E-04 | 7.9682E-04 | 1.6465E-03 | 7.1106E-03 | 1.4338E-02 | 4.5955E-03 | 1.8923E-02 |
| 6.600 | 1.1338E-02 | 4.5017E-04 | 1.6246E-04 | 3.2676E-04 | 8.0054E-04 | 2.3909E-03 | 3.0764E-03 | 8.3866E-04 | 8.6587E-03 |
| 6.700 | 5.3042E-03 | 2.1009E-04 | 1.2610E-04 | 1.8657E-04 | 5.8629E-04 | 1.1263E-03 | 2.5118E-03 | 5.4126E-03 | 3.3502E-03 |
| 6.800 | 6.8940E-02 | 3.0100E-03 | 1.5409E-03 | 4.8431E-04 | 4.6377E-04 | 4.0645E-03 | 2.5434E-02 | 3.1321E-02 | 3.7079E-02 |
| 6.900 | 7.6096E-02 | 3.2420E-03 | 1.7043E-03 | 5.7152E-04 | 4.9252E-04 | 4.7074E-03 | 2.7823E-02 | 3.2407E-02 | 4.1874E-02 |
| 7.000 | 7.6588E-02 | 3.3661E-03 | 1.7159E-03 | 5.7959E-04 | 4.9602E-04 | 4.7653E-03 | 2.7971E-02 | 3.3479E-02 | 4.2255E-02 |
| 7.100 | 7.6972E-02 | 3.3853E-03 | 1.7248E-03 | 5.8659E-04 | 4.9946E-04 | 4.8144E-03 | 2.8080E-02 | 3.3509E-02 | 4.2578E-02 |
| 7.200 | 3.2799E-02 | 1.5195E-03 | 7.6751E-04 | 3.5799E-04 | 3.6342E-04 | 2.6157E-03 | 1.1828E-02 | 1.0136E-02 | 2.0806E-02 |
| 7.300 | 2.0261E-02 | 9.8153E-04 | 5.0949E-04 | 2.1833E-04 | 4.6848E-04 | 1.6539E-03 | 6.9037E-03 | 3.3867E-03 | 1.3238E-02 |
| 7.400 | 1.3692E-02 | 6.9724E-04 | 3.9551E-04 | 1.0743E-04 | 4.8933E-04 | 9.6226E-04 | 5.0983E-03 | 4.6548E-03 | 8.0210E-03 |
| 7.500 | 1.2236E-02 | 5.8931E-04 | 2.9894E-04 | 4.8850E-04 | 2.5325E-03 | 1.5641E-03 | 3.7682E-03 | 2.5197E-02 | 1.4246E-02 |
| 7.600 | 3.0490E-02 | 1.4277E-03 | 4.4810E-04 | 1.0833E-03 | 4.7835E-03 | 3.7606E-03 | 4.3405E-03 | 4.7195E-02 | 3.5425E-02 |
| 7.700 | 3.3768E-02 | 1.5876E-03 | 4.9380E-04 | 1.1614E-03 | 5.0224E-03 | 4.0718E-03 | 4.6136E-03 | 4.6455E-02 | 3.8603E-02 |
| 7.800 | 3.4159E-02 | 1.6070E-03 | 4.9971E-04 | 1.1697E-03 | 5.0440E-03 | 4.1057E-03 | 4.6501E-03 | 4.5651E-02 | 3.8962E-02 |

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|------------|------------|------------|------------|------------|------------|------------|------------|------------|
| 155.000 | 3.4874E-04 | 1.6427E-03 | 5.1096E-04 | 1.1837E-03 | 5.0757E-03 | 4.1653E-03 | 4.7209E-03 | 4.5936E-02 | 3.9604E-02 |
| 160.000 | 3.4402E-04 | 1.5746E-03 | 5.3426E-04 | 9.3189E-04 | 3.5905E-03 | 3.3743E-03 | 4.7566E-03 | 3.4722E-02 | 3.3125E-02 |
| 130.000 | 1.9506E-04 | 1.0089E-04 | 3.7608E-04 | 4.4653E-04 | 1.4662E-03 | 1.2308E-03 | 2.7849E-03 | 1.2969E-02 | 1.8277E-02 |
| 200.000 | 1.8506E-04 | 8.8285E-04 | 3.0332E-04 | 3.3097E-04 | 1.0803E-03 | 4.1713E-04 | 1.1389E-03 | 9.2277E-03 | 1.4444E-02 |
| 220.000 | 1.3924E-04 | 8.0884E-04 | 2.1048E-04 | 3.3225E-04 | 8.4533E-04 | 3.2229E-04 | 2.6854E-03 | 4.0329E-03 | 1.0763E-02 |
| 240.000 | 1.3554E-04 | 7.5421E-04 | 4.1691E-04 | 3.5455E-04 | 7.2441E-04 | 1.9141E-02 | 1.9510E-02 | 2.2740E-02 | 1.7961E-02 |
| 242.000 | 1.6391E-02 | 7.9821E-04 | 5.4052E-04 | 4.6181E-04 | 8.4200E-04 | 2.2284E-02 | 2.2623E-02 | 2.8331E-02 | 2.3641E-02 |
| 242.100 | 1.6538E-02 | 8.0106E-04 | 5.4223E-04 | 4.6888E-04 | 8.4877E-04 | 2.2406E-02 | 2.2740E-02 | 2.8576E-02 | 2.3314E-02 |
| 242.200 | 1.6694E-02 | 8.0391E-04 | 5.5184E-04 | 4.7186E-04 | 8.5532E-04 | 2.2522E-02 | 2.2859E-02 | 2.8815E-02 | 2.4183E-02 |
| 243.000 | 1.7414E-04 | 8.2766E-04 | 5.9241E-04 | 5.0833E-04 | 9.0755E-04 | 2.3253E-02 | 2.3539E-02 | 3.0470E-02 | 2.6163E-02 |
| 250.000 | 2.0650E-02 | 9.0733E-04 | 5.8190E-04 | 5.1413E-04 | 9.9639E-04 | 1.7733E-02 | 1.7342E-02 | 2.7386E-02 | 2.7610E-02 |
| 260.000 | 1.8066E-02 | 8.6101E-04 | 4.0931E-04 | 3.6877E-04 | 8.0997E-04 | 1.0530E-02 | 9.3493E-03 | 1.8717E-02 | 2.1452E-02 |
| 280.000 | 1.5560E-02 | 8.1493E-04 | 2.3609E-04 | 2.1273E-04 | 5.2604E-04 | 5.4322E-03 | 3.5687E-03 | 1.3004E-02 | 1.6632E-02 |
| 300.000 | 1.3905E-02 | 7.9107E-04 | 1.0659E-04 | 4.9877E-04 | 2.4463E-04 | 5.1141E-03 | 2.7847E-03 | 1.0348E-02 | 1.3523E-02 |
| 320.000 | 1.1495E-02 | 7.7036E-04 | 4.9877E-04 | 4.8335E-04 | 1.0388E-03 | 4.1170E-03 | 1.2964E-02 | 7.6147E-03 | 9.5365E-03 |
| 340.000 | 1.9518E-02 | 7.7241E-04 | 2.5427E-03 | 2.4242E-03 | 5.6982E-03 | 4.4965E-03 | 5.5096E-02 | 1.6381E-02 | 2.9328E-02 |
| 340.600 | 2.0334E-02 | 7.7541E-04 | 2.5943E-03 | 2.4726E-03 | 5.8167E-03 | 4.6102E-03 | 5.6020E-02 | 1.7067E-02 | 3.0519E-02 |
| 340.700 | 2.0425E-02 | 7.7591E-04 | 2.6018E-03 | 2.4796E-03 | 5.8339E-03 | 4.6286E-03 | 5.6145E-02 | 1.7176E-02 | 3.0705E-02 |
| 341.000 | 2.0845E-02 | 7.7746E-04 | 2.6216E-03 | 2.4982E-03 | 5.8794E-03 | 4.6842E-03 | 5.6486E-02 | 1.7490E-02 | 3.1244E-02 |
| 350.000 | 2.3452E-02 | 7.9450E-04 | 2.1639E-03 | 2.0548E-03 | 4.8824E-03 | 5.1594E-03 | 4.4594E-02 | 1.8878E-02 | 3.2643E-02 |
| 330.000 | 1.8293E-02 | 7.7407E-04 | 1.0486E-03 | 9.8909E-04 | 2.3929E-03 | 4.2791E-03 | 1.9726E-02 | 1.3466E-02 | 2.2370E-02 |
| 410.000 | 1.6903E-02 | 7.6359E-04 | 8.1770E-04 | 7.6853E-04 | 1.8762E-03 | 3.9401E-03 | 1.4616E-02 | 1.1992E-02 | 1.5853E-02 |
| 440.000 | 1.6264E-02 | 7.5700E-04 | 7.3211E-04 | 5.7110E-04 | 1.6619E-03 | 3.7489E-03 | 1.2485E-02 | 1.1289E-02 | 1.3739E-02 |
| 455.000 | 1.6054E-02 | 7.5445E-04 | 6.9268E-04 | 6.4892E-04 | 1.5958E-03 | 3.5795E-03 | 1.1825E-02 | 1.1054E-02 | 1.3532E-02 |
| 457.900 | 1.6020E-02 | 7.5402E-04 | 6.8801E-04 | 6.4445E-04 | 1.5854E-03 | 3.5680E-03 | 1.1720E-02 | 1.1019E-02 | 1.3324E-02 |
| 460.000 | 1.5919E-02 | 7.5400E-04 | 6.8765E-04 | 6.4429E-04 | 1.5853E-03 | 3.5674E-03 | 1.1716E-02 | 1.1014E-02 | 1.3322E-02 |
| 475.000 | 1.5836E-02 | 7.5369E-04 | 6.8449E-04 | 6.4107E-04 | 1.5775E-03 | 3.5586E-03 | 1.1639E-02 | 1.0986E-02 | 1.3280E-02 |
| 477.000 | 1.5816E-02 | 7.5348E-04 | 6.8344E-04 | 6.4094E-04 | 1.5702E-03 | 3.5504E-03 | 1.1638E-02 | 1.0983E-02 | 1.3013E-02 |
| 430.000 | 1.5790E-02 | 7.5131E-04 | 6.6103E-04 | 6.1859E-04 | 1.5247E-03 | 3.5972E-03 | 1.1111E-02 | 1.0787E-02 | 1.7987E-02 |
| 550.000 | 1.5349E-02 | 7.5093E-04 | 6.5748E-04 | 6.1518E-04 | 1.5167E-03 | 3.5876E-03 | 1.1031E-02 | 1.0756E-02 | 1.7942E-02 |
| 600.000 | 1.5144E-02 | 7.4411E-04 | 6.0331E-04 | 5.6315E-04 | 1.3945E-03 | 3.4230E-03 | 9.8044E-03 | 1.0251E-02 | 1.7230E-02 |
| 612.000 | 1.5126E-02 | 7.4087E-04 | 5.8258E-04 | 5.4319E-04 | 1.3475E-03 | 3.3504E-03 | 9.3318E-03 | 1.0038E-02 | 1.6943E-02 |
| 615.000 | 1.5126E-02 | 7.4022E-04 | 5.7877E-04 | 5.3951E-04 | 1.3388E-03 | 3.3362E-03 | 9.2440E-03 | 9.5973E-03 | 1.6399E-02 |
| 616.000 | 1.5126E-02 | 7.4007E-04 | 5.7786E-04 | 5.3864E-04 | 1.3368E-03 | 3.3299E-03 | 9.2231E-03 | 9.5877E-03 | 1.6876E-02 |
| 650.000 | 1.5117E-02 | 7.4001E-04 | 5.7757E-04 | 5.3836E-04 | 1.3361E-03 | 3.3200E-03 | 9.2171E-03 | 9.9845E-03 | 1.6872E-02 |
| 650.000 | 1.5033E-02 | 7.3843E-04 | 5.6869E-04 | 5.2979E-04 | 1.3159E-03 | 3.2981E-03 | 9.0133E-03 | 9.8877E-03 | 1.6745E-02 |

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 3.000 | 25.15 | 25.81 | 25.97 | 25.57 | 23.57 | 19.86 | 12.56 | 4.08 | 9.86 |
| 3.100 | 49.69 | 50.38 | 50.53 | 50.07 | 47.73 | 42.79 | 29.75 | 6.43 | 23.49 |
| 3.200 | 113.36 | 114.18 | 113.36 | 113.88 | 111.17 | 104.20 | 76.24 | 8.54 | 54.00 |
| 4.000 | 173.08 | 174.91 | 175.23 | 176.30 | 157.51 | 131.41 | 4.49 | 3.31 | 3.77 |
| 6.000 | 33.21 | 177.00 | 178.77 | 169.23 | 6.46 | 5.05 | 4.59 | 4.18 | 3.81 |
| 8.000 | 11.32 | 79.29 | 179.05 | 15.55 | 10.22 | 9.67 | 9.35 | 8.52 | 7.11 |
| 9.000 | 26.25 | 35.09 | 177.31 | 28.95 | 26.28 | 25.82 | 25.47 | 24.18 | 21.31 |
| 9.400 | 74.77 | 81.23 | 169.66 | 77.37 | 75.22 | 74.77 | 76.40 | 72.79 | 64.64 |
| 9.500 | 46.69 | 104.82 | 169.67 | 101.27 | 99.22 | 98.78 | 98.38 | 92.08 | 82.08 |
| 10.000 | 156.45 | 161.53 | 177.74 | 159.14 | 157.48 | 157.06 | 150.59 | 154.69 | 146.37 |
| 12.000 | 171.38 | 177.04 | 179.03 | 176.19 | 173.24 | 174.33 | 173.90 | 164.92 | 152.23 |
| 14.000 | 158.21 | 178.33 | 178.64 | 177.93 | 177.31 | 176.49 | 176.57 | 29.12 | 6.95 |
| 16.000 | 20.48 | 178.37 | 177.50 | 178.24 | 177.90 | 176.30 | 169.72 | 10.56 | 6.62 |
| 18.000 | 11.39 | 177.21 | 173.53 | 177.55 | 177.92 | 173.61 | 80.60 | 10.03 | 8.15 |
| 20.000 | 13.55 | 168.91 | 126.29 | 172.21 | 177.32 | 147.16 | 22.36 | 14.43 | 13.17 |
| 22.000 | 44.98 | 80.54 | 60.47 | 92.53 | 170.49 | 62.47 | 50.54 | 48.04 | 46.97 |
| 22.400 | 75.05 | 101.06 | 88.12 | 107.12 | 165.56 | 89.01 | 80.65 | 78.71 | 77.64 |
| 22.500 | 84.96 | 109.46 | 97.61 | 114.68 | 165.26 | 88.28 | 90.60 | 88.78 | 87.71 |
| 23.000 | 127.06 | 146.62 | 138.23 | 149.37 | 170.29 | 138.11 | 133.03 | 131.80 | 130.73 |
| 25.000 | 146.34 | 176.75 | 169.46 | 177.03 | 171.10 | 165.64 | 169.87 | 171.74 | 170.26 |
| 30.000 | 16.72 | 178.60 | 78.16 | 178.44 | 47.74 | 34.31 | 159.59 | 175.94 | 171.24 |
| 33.000 | 15.59 | 178.58 | 23.74 | 177.76 | 22.01 | 19.56 | 40.91 | 176.01 | 145.02 |
| 36.000 | 34.77 | 175.61 | 39.03 | 167.01 | 38.50 | 36.97 | 40.01 | 165.48 | 52.26 |
| 37.000 | 66.57 | 169.39 | 148.63 | 69.96 | 68.50 | 68.50 | 70.07 | 147.05 | 78.22 |
| 37.400 | 89.60 | 168.03 | 93.29 | 149.17 | 92.90 | 91.44 | 92.56 | 147.66 | 99.84 |
| 37.500 | 95.85 | 168.28 | 99.52 | 150.75 | 99.13 | 97.68 | 98.69 | 149.24 | 105.78 |
| 38.000 | 122.90 | 171.59 | 126.45 | 160.84 | 126.09 | 124.64 | 125.17 | 159.50 | 131.57 |
| 40.000 | 160.55 | 177.66 | 163.88 | 175.72 | 163.56 | 162.06 | 160.97 | 174.65 | 166.46 |
| 45.000 | 171.27 | 178.33 | 175.36 | 178.16 | 174.95 | 172.78 | 165.98 | 177.32 | 175.96 |
| 50.000 | 169.72 | 177.58 | 177.38 | 177.73 | 176.71 | 172.50 | 104.12 | 177.03 | 177.30 |
| 60.000 | 30.66 | 168.89 | 178.19 | 170.75 | 175.32 | 69.17 | 12.12 | 171.65 | 176.61 |
| 70.000 | 19.72 | 33.16 | 177.03 | 36.83 | 102.89 | 21.86 | 17.07 | 42.99 | 151.47 |
| 76.000 | 68.32 | 71.27 | 163.87 | 72.84 | 79.32 | 68.01 | 54.55 | 69.14 | 86.29 |
| 76.800 | 88.39 | 92.92 | 162.10 | 94.47 | 99.89 | 90.10 | 86.68 | 90.28 | 105.01 |
| 76.900 | 91.41 | 95.90 | 162.26 | 97.43 | 102.76 | 93.13 | 89.70 | 93.16 | 107.69 |
| 77.000 | 94.38 | 98.82 | 162.39 | 100.37 | 105.55 | 96.10 | 92.62 | 95.99 | 110.32 |
| 80.000 | 148.93 | 152.29 | 172.53 | 154.04 | 157.12 | 150.87 | 147.26 | 146.35 | 158.71 |
| 90.000 | 164.80 | 167.05 | 166.65 | 173.17 | 175.12 | 171.11 | 160.23 | 64.42 | 171.82 |
| 100.000 | 64.86 | 71.93 | 47.97 | 166.93 | 175.59 | 160.98 | 43.87 | 21.85 | 130.70 |
| 110.000 | 81.55 | 82.77 | 81.77 | 101.27 | 159.18 | 94.63 | 79.55 | 74.33 | 86.80 |
| 110.800 | 97.04 | 98.20 | 97.39 | 113.21 | 158.67 | 107.90 | 95.24 | 90.17 | 101.66 |
| 110.900 | 99.06 | 100.22 | 99.43 | 114.85 | 158.78 | 109.68 | 97.28 | 92.22 | 103.61 |
| 111.000 | 101.07 | 102.22 | 101.45 | 116.49 | 158.94 | 111.46 | 99.32 | 94.27 | 105.55 |
| 120.000 | 167.33 | 168.16 | 168.34 | 170.10 | 169.20 | 169.29 | 166.71 | 160.09 | 168.72 |
| 130.000 | 174.91 | 173.72 | 174.63 | 170.15 | 132.35 | 171.87 | 173.41 | 148.20 | 172.39 |
| 140.000 | 169.02 | 170.54 | 174.50 | 137.13 | 36.54 | 160.36 | 174.15 | 43.42 | 161.22 |
| 150.000 | 108.91 | 116.84 | 155.24 | 62.05 | 48.36 | 77.54 | 164.64 | 48.26 | 78.80 |
| 154.000 | 112.06 | 115.42 | 139.13 | 94.05 | 86.39 | 99.32 | 152.01 | 85.84 | 101.24 |
| 154.700 | 119.09 | 121.97 | 141.73 | 103.50 | 96.48 | 107.87 | 152.73 | 95.89 | 109.88 |
| 154.800 | 120.16 | 122.99 | 142.23 | 104.88 | 97.94 | 109.13 | 152.94 | 97.35 | 111.15 |

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 155.000 | 124.54 | 125.05 | 143.31 | 107.64 | 100.87 | 111.67 | 153.44 | 100.26 | 113.71 |
| 160.000 | 160.44 | 161.71 | 167.35 | 153.86 | 149.88 | 154.25 | 168.74 | 149.05 | 158.95 |
| 170.000 | 170.86 | 177.44 | 177.16 | 175.21 | 174.45 | 167.36 | 172.69 | 173.09 | 176.14 |
| 200.000 | 177.84 | 178.62 | 176.49 | 176.03 | 175.56 | 77.90 | 131.77 | 173.20 | 176.83 |
| 220.000 | 176.44 | 178.48 | 168.89 | 170.64 | 175.44 | 26.27 | 34.49 | 150.91 | 171.66 |
| 240.000 | 144.06 | 168.51 | 98.00 | 105.26 | 147.46 | 67.12 | 56.68 | 81.74 | 106.84 |
| 242.000 | 142.71 | 166.49 | 107.61 | 112.74 | 145.32 | 83.38 | 82.53 | 95.73 | 114.50 |
| 242.100 | 142.92 | 166.45 | 108.24 | 113.28 | 145.37 | 84.29 | 83.41 | 96.53 | 115.04 |
| 242.200 | 142.75 | 166.41 | 108.87 | 113.83 | 145.44 | 85.19 | 84.30 | 97.34 | 115.60 |
| 243.000 | 144.50 | 166.36 | 114.23 | 118.57 | 146.48 | 92.61 | 91.56 | 104.02 | 120.42 |
| 250.000 | 160.24 | 174.11 | 153.56 | 155.13 | 164.98 | 142.89 | 140.43 | 149.93 | 157.56 |
| 250.000 | 175.25 | 178.07 | 167.76 | 168.21 | 171.96 | 164.16 | 159.19 | 168.19 | 171.82 |
| 280.000 | 177.72 | 179.25 | 166.21 | 165.73 | 167.97 | 173.46 | 153.97 | 174.81 | 176.04 |
| 300.000 | 177.06 | 179.44 | 111.06 | 105.21 | 117.49 | 175.58 | 56.08 | 174.76 | 174.54 |
| 320.000 | 170.26 | 179.23 | 42.81 | 41.96 | 44.04 | 174.13 | 35.01 | 164.60 | 157.83 |
| 340.000 | 135.32 | 176.38 | 93.55 | 93.30 | 93.96 | 155.31 | 90.58 | 125.49 | 118.11 |
| 340.600 | 136.83 | 176.35 | 97.44 | 97.20 | 97.85 | 155.65 | 94.53 | 127.64 | 120.78 |
| 340.700 | 137.11 | 176.34 | 98.10 | 97.86 | 98.50 | 155.73 | 95.20 | 128.02 | 121.24 |
| 341.000 | 137.97 | 176.34 | 100.07 | 99.83 | 100.47 | 155.98 | 97.20 | 129.17 | 122.64 |
| 350.000 | 162.54 | 178.19 | 145.16 | 144.99 | 145.45 | 169.28 | 142.97 | 159.64 | 157.20 |
| 380.000 | 177.63 | 179.66 | 172.91 | 172.83 | 173.05 | 177.87 | 171.73 | 176.67 | 176.44 |
| 410.000 | 176.98 | 179.80 | 176.73 | 176.68 | 176.81 | 178.80 | 175.95 | 178.45 | 178.47 |
| 440.000 | 179.39 | 179.85 | 178.04 | 178.00 | 178.09 | 179.15 | 177.47 | 179.02 | 179.09 |
| 440.000 | 179.50 | 179.87 | 178.40 | 178.36 | 178.44 | 179.26 | 177.90 | 179.18 | 179.26 |
| 457.800 | 174.51 | 179.87 | 178.42 | 178.42 | 178.49 | 179.28 | 177.96 | 179.20 | 179.28 |
| 457.900 | 174.51 | 179.87 | 178.45 | 178.42 | 178.49 | 179.28 | 177.96 | 179.20 | 179.28 |
| 460.000 | 174.52 | 179.87 | 178.49 | 178.46 | 178.53 | 179.29 | 177.96 | 179.20 | 179.28 |
| 470.000 | 174.59 | 179.88 | 178.73 | 178.70 | 178.76 | 179.37 | 178.01 | 179.22 | 179.30 |
| 477.000 | 174.60 | 179.88 | 178.76 | 178.73 | 178.79 | 179.38 | 178.30 | 179.33 | 179.41 |
| 480.000 | 174.61 | 179.89 | 178.80 | 178.77 | 178.82 | 179.39 | 178.33 | 179.34 | 179.42 |
| 500.000 | 174.77 | 179.92 | 179.33 | 179.31 | 179.35 | 179.39 | 178.38 | 179.36 | 179.44 |
| 550.000 | 174.85 | 179.94 | 179.51 | 179.50 | 179.52 | 179.51 | 179.07 | 179.61 | 179.68 |
| 600.000 | 174.85 | 179.94 | 179.55 | 179.53 | 179.55 | 179.52 | 179.31 | 179.70 | 179.76 |
| 612.000 | 174.84 | 179.94 | 179.55 | 179.54 | 179.55 | 179.71 | 179.35 | 179.72 | 179.78 |
| 615.000 | 174.84 | 179.94 | 179.55 | 179.54 | 179.55 | 179.71 | 179.36 | 179.72 | 179.78 |
| 616.000 | 174.84 | 179.94 | 179.55 | 179.54 | 179.56 | 179.71 | 179.36 | 179.72 | 179.78 |
| 650.000 | 174.86 | 179.95 | 179.63 | 179.61 | 179.63 | 179.75 | 179.36 | 179.76 | 179.82 |

TRANSFER RESPONSE, ROW 5 AMP IN G'S AND PHASE IN DEG

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|------------|------------|------------|------------|------------|------------|------------|------------|------------|
| 3.000 | 9.1107E-04 | 6.5303E-04 | 5.3022E-04 | 4.1245E-04 | 2.9621E-04 | 2.2312E-04 | 1.5093E-04 | 1.7764E-05 | 1.0919E-04 |
| 3.100 | 1.6012E-03 | 1.2939E-03 | 1.0188E-03 | 7.7551E-04 | 5.3771E-04 | 3.9062E-04 | 2.4814E-04 | 5.0174E-05 | 2.5116E-04 |
| 3.200 | 2.0410E-03 | 1.6311E-03 | 1.2628E-03 | 9.3802E-04 | 6.2369E-04 | 4.3215E-04 | 2.5071E-04 | 1.4501E-04 | 3.6898E-04 |
| 4.000 | 3.6265E-04 | 2.6489E-04 | 1.7574E-04 | 9.8715E-05 | 2.9005E-05 | 1.6145E-05 | 2.3361E-04 | 9.2836E-05 | 1.3112E-04 |
| 6.000 | 3.9358E-04 | 2.3132E-04 | 6.4274E-05 | 3.8821E-05 | 1.4314E-05 | 1.9616E-04 | 2.7001E-04 | 2.8113E-04 | 2.8113E-04 |
| 8.000 | 1.0597E-03 | 5.1617E-04 | 4.2748E-05 | 3.3302E-04 | 6.3975E-04 | 7.8957E-04 | 8.8956E-04 | 9.6713E-04 | 9.7586E-04 |
| 9.000 | 3.2737E-03 | 1.4768E-03 | 8.2376E-04 | 1.2470E-03 | 2.1912E-03 | 3.6883E-03 | 3.1888E-03 | 3.1888E-03 | 3.2219E-03 |
| 9.400 | 7.8021E-03 | 3.4164E-03 | 3.4365E-04 | 3.1333E-03 | 5.3654E-03 | 6.4442E-03 | 7.1643E-03 | 7.7388E-03 | 8.0633E-03 |
| 9.500 | 8.0081E-03 | 3.4811E-03 | 3.8907E-04 | 3.2513E-03 | 5.5348E-03 | 6.6377E-03 | 7.3741E-03 | 7.5587E-03 | 8.0633E-03 |
| 10.000 | 3.4166E-03 | 1.4318E-03 | 2.4099E-04 | 2.4121E-03 | 3.8728E-03 | 2.8728E-03 | 3.1825E-03 | 3.4386E-03 | 3.4992E-03 |
| 12.000 | 1.0204E-03 | 3.7851E-04 | 1.4002E-04 | 4.8349E-04 | 7.3705E-04 | 8.5872E-04 | 9.4456E-04 | 1.0343E-03 | 1.0793E-03 |
| 14.000 | 7.8977E-04 | 2.4518E-04 | 1.3274E-04 | 3.5422E-04 | 5.0357E-04 | 5.7523E-04 | 6.3046E-04 | 7.0813E-04 | 7.6601E-04 |
| 16.000 | 6.8206E-04 | 1.9931E-04 | 1.3774E-04 | 3.0563E-04 | 4.0319E-04 | 4.4970E-04 | 4.9089E-04 | 5.6951E-04 | 6.4260E-04 |
| 18.000 | 7.0037E-04 | 1.8342E-04 | 1.5079E-04 | 2.8363E-04 | 3.3997E-04 | 3.6498E-04 | 3.9334E-04 | 4.6889E-04 | 5.5049E-04 |
| 20.000 | 8.0690E-04 | 1.9016E-04 | 1.7629E-04 | 2.7858E-04 | 2.8703E-04 | 2.8236E-04 | 2.8686E-04 | 3.2655E-04 | 3.8109E-04 |
| 22.000 | 1.2317E-03 | 2.6819E-04 | 2.5066E-04 | 3.1131E-04 | 2.2899E-04 | 2.0623E-04 | 2.9555E-04 | 7.2572E-04 | 1.2274E-03 |
| 22.400 | 1.2548E-03 | 2.7146E-04 | 2.4771E-04 | 2.5944E-04 | 2.3720E-04 | 3.2355E-04 | 5.5466E-04 | 1.3478E-03 | 2.2487E-03 |
| 22.500 | 1.1881E-03 | 2.5650E-04 | 2.3372E-04 | 2.7675E-04 | 2.4346E-04 | 3.5840E-04 | 6.1434E-04 | 1.4802E-03 | 2.4639E-03 |
| 23.000 | 6.2419E-04 | 1.2798E-04 | 1.3878E-04 | 1.8501E-04 | 2.5339E-04 | 3.9774E-04 | 6.5177E-04 | 1.5189E-03 | 2.5185E-03 |
| 26.000 | 7.7262E-04 | 1.1139E-04 | 1.9688E-04 | 1.9644E-04 | 1.5005E-04 | 1.7467E-04 | 2.9293E-04 | 8.5239E-04 | 1.5563E-03 |
| 30.000 | 1.4621E-03 | 1.4792E-04 | 3.3173E-04 | 1.7852E-04 | 7.2951E-05 | 1.2302E-04 | 6.9362E-05 | 8.3850E-04 | 1.9391E-03 |
| 33.000 | 2.5692E-03 | 1.9527E-04 | 5.7178E-04 | 1.3381E-04 | 4.1703E-04 | 5.7816E-04 | 4.1572E-04 | 1.0219E-03 | 3.0023E-03 |
| 36.000 | 7.5529E-03 | 3.7108E-04 | 1.5783E-03 | 1.8713E-04 | 1.9071E-03 | 2.4963E-03 | 2.2179E-03 | 1.9002E-03 | 7.6616E-03 |
| 37.000 | 1.3062E-02 | 5.3131E-04 | 2.6672E-03 | 5.3036E-04 | 3.6159E-04 | 4.6777E-03 | 4.0730E-03 | 2.7512E-03 | 1.2849E-02 |
| 37.400 | 1.4593E-02 | 5.4735E-04 | 2.9520E-03 | 6.8820E-04 | 4.1659E-03 | 5.3682E-03 | 4.7130E-03 | 2.6638E-03 | 1.3904E-02 |
| 37.500 | 1.4600E-02 | 5.3378E-04 | 2.9461E-03 | 7.1724E-04 | 4.2044E-03 | 5.4108E-03 | 4.7608E-03 | 2.7968E-03 | 1.3820E-02 |
| 38.000 | 1.2958E-02 | 4.1155E-04 | 2.5233E-03 | 4.8805E-04 | 3.7772E-03 | 4.8374E-03 | 4.2956E-03 | 2.1928E-03 | 1.1707E-02 |
| 40.000 | 5.4257E-03 | 9.3972E-05 | 1.0271E-03 | 4.8805E-04 | 1.8087E-03 | 2.2767E-03 | 2.0772E-03 | 5.2344E-04 | 4.3625E-03 |
| 45.000 | 2.8971E-03 | 5.3370E-05 | 3.8669E-04 | 3.6817E-04 | 9.0052E-04 | 1.0919E-03 | 1.0276E-03 | 4.1419E-05 | 1.3469E-03 |
| 50.000 | 1.8381E-03 | 9.8141E-05 | 2.4696E-04 | 3.5695E-04 | 6.7252E-04 | 7.7999E-04 | 3.1474E-04 | 1.4522E-04 | 8.5445E-04 |
| 60.000 | 1.9035E-03 | 2.0085E-04 | 1.4278E-04 | 4.0743E-04 | 4.3294E-04 | 3.9818E-04 | 3.1474E-04 | 1.1810E-04 | 2.2431E-04 |
| 70.000 | 3.2468E-04 | 5.4541E-04 | 6.4704E-05 | 6.5933E-04 | 1.5579E-04 | 5.4756E-04 | 9.5967E-04 | 4.5565E-04 | 1.0120E-03 |
| 76.000 | 1.1922E-02 | 1.9875E-03 | 2.8317E-04 | 1.5932E-03 | 1.8213E-03 | 4.3385E-03 | 6.4800E-03 | 3.4941E-03 | 6.0331E-03 |
| 76.800 | 1.3116E-02 | 2.2192E-03 | 3.6565E-04 | 1.6701E-03 | 2.2476E-03 | 5.1662E-03 | 7.6643E-03 | 4.1967E-03 | 7.1303E-03 |
| 76.900 | 1.3124E-02 | 2.2240E-03 | 3.7182E-04 | 1.6622E-03 | 2.2760E-03 | 5.2147E-03 | 7.7321E-03 | 4.2406E-03 | 7.1569E-03 |
| 77.000 | 1.3101E-02 | 2.2254E-03 | 3.7996E-04 | 1.5464E-03 | 2.3070E-03 | 5.2590E-03 | 7.7898E-03 | 4.2822E-03 | 7.2007E-03 |
| 80.000 | 6.7084E-03 | 1.2000E-03 | 3.0183E-04 | 6.7592E-04 | 1.6465E-03 | 3.4525E-03 | 5.0526E-03 | 2.5367E-03 | 4.5698E-03 |
| 90.000 | 1.7128E-03 | 1.5247E-04 | 1.6398E-04 | 4.0535E-05 | 8.0054E-04 | 1.5729E-03 | 2.3777E-03 | 1.6498E-03 | 2.1349E-03 |
| 100.000 | 7.3521E-04 | 1.7287E-04 | 1.0351E-04 | 9.8857E-05 | 5.8629E-04 | 1.2858E-03 | 1.7978E-03 | 2.6969E-03 | 2.9155E-03 |
| 110.000 | 2.9321E-03 | 6.1794E-04 | 4.3965E-04 | 3.3961E-04 | 4.6377E-04 | 1.3176E-03 | 2.7851E-03 | 2.8883E-03 | 3.1268E-03 |
| 110.300 | 3.4397E-03 | 7.3005E-04 | 5.2166E-04 | 3.3961E-04 | 4.3252E-04 | 1.1793E-03 | 2.5493E-03 | 2.6969E-03 | 2.8155E-03 |
| 113.900 | 3.4667E-03 | 7.4684E-04 | 5.2964E-04 | 3.4725E-04 | 4.3602E-04 | 1.1563E-03 | 2.5057E-03 | 2.6574E-03 | 2.8119E-03 |
| 114.000 | 3.5280E-03 | 7.5044E-04 | 5.3678E-04 | 3.5443E-04 | 4.9945E-04 | 1.1324E-03 | 2.4591E-03 | 2.6142E-03 | 2.8244E-03 |
| 120.000 | 2.3495E-03 | 5.2866E-04 | 3.8778E-04 | 3.9508E-04 | 3.6342E-04 | 5.0340E-04 | 1.0123E-03 | 1.1847E-03 | 1.0966E-03 |
| 130.000 | 2.3050E-03 | 5.2557E-04 | 3.7480E-04 | 4.7962E-04 | 1.6684E-04 | 3.8133E-04 | 1.3368E-03 | 2.0788E-03 | 1.8817E-03 |
| 140.000 | 7.5621E-03 | 7.2879E-04 | 4.7273E-04 | 7.1246E-04 | 4.8933E-04 | 2.5124E-04 | 1.6813E-03 | 3.6844E-03 | 3.2459E-03 |
| 150.000 | 7.5601E-03 | 1.6514E-03 | 8.9545E-04 | 1.5861E-03 | 2.9326E-03 | 2.0071E-03 | 2.6819E-03 | 9.6054E-03 | 8.2911E-03 |
| 154.000 | 1.1571E-02 | 2.3526E-03 | 1.2347E-03 | 2.3575E-03 | 4.1584E-03 | 3.3220E-03 | 3.3220E-03 | 1.5284E-02 | 1.3114E-02 |
| 154.700 | 1.1986E-02 | 2.5683E-03 | 1.2294E-03 | 2.3808E-03 | 5.0224E-03 | 4.4183E-03 | 3.2215E-03 | 1.5628E-02 | 1.3396E-02 |
| 154.900 | 1.1987E-02 | 2.5674E-03 | 1.2252E-03 | 2.3377E-03 | 5.0440E-03 | 4.4453E-03 | 3.1972E-03 | 1.5634E-02 | 1.3400E-02 |

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|------------|------------|------------|------------|------------|------------|------------|------------|------------|
| 155.000 | 1.1904E-04 | 4.5800E-03 | 1.2143E-03 | 2.3664E-03 | 5.0757E-03 | 4.4882E-03 | 3.1448E-03 | 1.5619E-02 | 1.3383E-02 |
| 160.000 | 7.0822E-04 | 1.4803E-03 | 5.8824E-04 | 1.2900E-03 | 3.5904E-03 | 3.4020E-03 | 1.1762E-03 | 9.3660E-03 | 7.9821E-03 |
| 180.000 | 1.9980E-03 | 3.7208E-04 | 3.9560E-05 | 1.9163E-04 | 1.8662E-03 | 1.5712E-03 | 5.5339E-04 | 2.4774E-03 | 2.1041E-03 |
| 200.000 | 1.4252E-03 | 2.3222E-04 | 1.3096E-04 | 3.9167E-05 | 1.0800E-03 | 1.1720E-03 | 7.1894E-04 | 1.3663E-03 | 1.1913E-03 |
| 220.000 | 1.4749E-03 | 2.1125E-04 | 2.6416E-04 | 1.6481E-04 | 8.4933E-04 | 7.6330E-04 | 6.1939E-04 | 7.0135E-04 | 6.6043E-04 |
| 240.000 | 2.5031E-03 | 3.3031E-04 | 5.6285E-04 | 4.8748E-04 | 7.2441E-04 | 2.4709E-03 | 2.3901E-03 | 2.7891E-03 | 2.4027E-03 |
| 242.000 | 2.5182E-03 | 3.3191E-04 | 5.7710E-04 | 5.0932E-04 | 9.4209E-04 | 3.1909E-03 | 3.1523E-03 | 3.5417E-03 | 3.0883E-03 |
| 242.100 | 2.5129E-03 | 3.3121E-04 | 5.7640E-04 | 5.0918E-04 | 8.6867E-04 | 3.2239E-03 | 3.1883E-03 | 3.5754E-03 | 3.1196E-03 |
| 242.200 | 2.5071E-03 | 3.3044E-04 | 5.7557E-04 | 5.0891E-04 | 8.5532E-04 | 3.2563E-03 | 3.2237E-03 | 3.6086E-03 | 3.1505E-03 |
| 243.000 | 2.4352E-03 | 3.2103E-04 | 5.6300E-04 | 5.0152E-04 | 9.0755E-04 | 3.4901E-03 | 3.4843E-03 | 3.8441E-03 | 3.3717E-03 |
| 250.000 | 1.0251E-03 | 1.3613E-04 | 2.5160E-04 | 2.3651E-04 | 9.9639E-04 | 3.4488E-03 | 3.7012E-03 | 3.6544E-03 | 3.3253E-03 |
| 253.000 | 4.6457E-04 | 4.0104E-05 | 1.4693E-04 | 1.2236E-04 | 8.0977E-04 | 2.5978E-03 | 3.0916E-03 | 2.6957E-03 | 2.5066E-03 |
| 280.000 | 9.3572E-04 | 7.1216E-05 | 3.2903E-04 | 2.9743E-04 | 5.2304E-04 | 2.1379E-03 | 3.1690E-03 | 2.2882E-03 | 2.3745E-03 |
| 300.000 | 1.5848E-03 | 1.1181E-04 | 6.0557E-04 | 5.7165E-04 | 2.463E-04 | 2.1410E-03 | 4.1478E-03 | 2.5831E-03 | 2.9341E-03 |
| 320.000 | 3.1482E-03 | 2.0147E-04 | 1.2865E-03 | 1.2412E-03 | 1.0888E-03 | 2.5507E-03 | 7.1094E-03 | 3.5063E-03 | 4.4857E-03 |
| 340.000 | 8.8619E-03 | 5.1442E-04 | 3.8169E-03 | 3.7270E-03 | 5.6982E-03 | 3.6098E-03 | 1.7896E-02 | 8.7974E-03 | 1.2221E-02 |
| 340.600 | 8.9272E-03 | 5.1673E-04 | 3.8504E-03 | 3.7607E-03 | 5.8167E-03 | 3.5423E-03 | 1.7972E-02 | 8.8071E-03 | 1.2274E-02 |
| 343.700 | 8.9339E-03 | 5.1688E-04 | 3.8542E-03 | 3.7646E-03 | 5.8339E-03 | 3.5292E-03 | 1.7977E-02 | 8.8044E-03 | 1.2277E-02 |
| 361.000 | 8.9458E-03 | 5.1681E-04 | 3.8619E-03 | 3.7724E-03 | 5.8794E-03 | 3.4890E-03 | 1.7976E-02 | 8.7891E-03 | 1.2276E-02 |
| 380.000 | 6.2381E-03 | 3.4347E-04 | 2.7464E-03 | 2.6919E-03 | 4.8824E-03 | 1.5337E-03 | 1.1986E-02 | 5.9929E-03 | 8.2037E-03 |
| 380.000 | 2.0583E-03 | 9.9906E-05 | 9.5513E-04 | 9.4194E-04 | 2.3929E-03 | 6.2630E-04 | 3.4469E-03 | 1.3870E-03 | 2.4221E-03 |
| 410.000 | 1.2600E-03 | 5.4388E-05 | 6.0750E-04 | 5.9997E-04 | 1.8762E-03 | 8.4948E-04 | 1.8566E-03 | 6.4699E-04 | 1.3665E-03 |
| 440.000 | 9.3919E-04 | 3.6542E-05 | 4.6594E-04 | 4.5974E-04 | 1.6619E-03 | 9.3815E-04 | 1.2237E-03 | 3.6775E-04 | 9.5664E-04 |
| 455.000 | 8.4226E-04 | 3.1259E-05 | 4.2274E-04 | 4.1571E-04 | 1.5958E-03 | 9.6323E-04 | 1.0334E-03 | 2.8717E-04 | 8.3575E-04 |
| 457.800 | 8.2698E-04 | 3.0432E-05 | 4.1566E-04 | 4.0965E-04 | 1.5850E-03 | 9.6716E-04 | 1.0024E-03 | 2.7467E-04 | 8.1682E-04 |
| 460.000 | 8.1544E-04 | 3.0403E-05 | 4.1076E-04 | 4.0476E-04 | 1.5775E-03 | 9.6721E-04 | 1.0024E-03 | 2.7422E-04 | 8.1617E-04 |
| 475.000 | 7.6723E-04 | 2.6154E-05 | 3.8009E-04 | 3.7408E-04 | 1.5302E-03 | 9.7000E-04 | 9.8093E-04 | 2.6528E-04 | 8.0263E-04 |
| 477.000 | 7.3935E-04 | 2.5734E-05 | 3.7653E-04 | 3.7052E-04 | 1.5247E-03 | 9.8858E-04 | 8.4717E-04 | 2.1058E-04 | 7.1901E-04 |
| 480.000 | 7.2788E-04 | 2.5125E-05 | 3.7136E-04 | 3.6533E-04 | 1.5167E-03 | 9.9131E-04 | 8.0934E-04 | 2.0431E-04 | 7.0941E-04 |
| 550.000 | 5.5634E-04 | 1.6235E-05 | 2.9310E-04 | 2.8640E-04 | 1.3945E-03 | 1.0286E-03 | 4.7414E-04 | 6.6816E-05 | 4.9194E-04 |
| 600.000 | 4.9203E-04 | 1.3078E-05 | 2.6348E-04 | 2.5622E-04 | 1.3475E-03 | 1.0403E-03 | 3.4968E-04 | 2.3699E-05 | 4.1917E-04 |
| 612.000 | 4.8108E-04 | 1.2516E-05 | 2.5804E-04 | 2.5068E-04 | 1.3388E-03 | 1.0423E-03 | 3.2716E-04 | 1.6575E-05 | 4.0616E-04 |
| 615.000 | 4.7834E-04 | 1.2384E-05 | 2.5675E-04 | 2.4993E-04 | 1.3368E-03 | 1.0428E-03 | 3.2172E-04 | 1.5001E-05 | 4.0309E-04 |
| 616.000 | 4.7745E-04 | 1.2341E-05 | 2.5636E-04 | 2.4894E-04 | 1.3361E-03 | 1.0429E-03 | 3.2001E-04 | 1.4502E-05 | 4.0208E-04 |
| 650.000 | 4.5075E-04 | 1.1062E-05 | 2.4377E-04 | 2.3603E-04 | 1.3159E-03 | 1.0472E-03 | 2.6780E-04 | 9.6514E-06 | 3.7238E-04 |

| HERTZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 3.000 | 27.10 | 26.64 | 25.97 | 25.04 | 23.57 | 22.09 | 19.61 | 309.57 | 214.63 |
| 3.100 | 51.61 | 51.13 | 50.43 | 49.42 | 47.73 | 45.90 | 42.52 | 275.15 | 238.13 |
| 3.200 | 115.59 | 114.33 | 113.07 | 112.00 | 111.17 | 108.77 | 103.65 | 319.03 | 301.47 |
| 4.000 | 177.05 | 176.41 | 175.18 | 172.41 | 157.51 | 36.71 | 9.83 | 2.44 | 0.42 |
| 6.000 | 182.59 | 181.73 | 178.25 | 15.01 | 6.46 | 5.54 | 5.07 | 4.53 | 4.21 |
| 8.000 | 188.71 | 187.76 | 169.16 | 11.54 | 10.22 | 9.93 | 9.77 | 9.64 | 9.62 |
| 9.000 | 205.34 | 204.37 | 53.44 | 27.17 | 26.28 | 26.06 | 25.92 | 25.92 | 25.98 |
| 9.400 | 258.44 | 253.52 | 89.11 | 76.02 | 75.22 | 75.03 | 74.94 | 74.94 | 75.03 |
| 9.500 | 278.51 | 277.53 | 111.75 | 100.01 | 99.22 | 99.02 | 98.93 | 98.94 | 99.03 |
| 10.000 | 336.99 | 336.00 | 165.43 | 158.20 | 157.48 | 157.30 | 157.23 | 157.27 | 157.41 |
| 12.000 | 354.49 | 354.45 | 178.73 | 175.86 | 175.24 | 175.08 | 175.04 | 175.21 | 175.47 |
| 14.000 | 358.27 | 357.14 | 180.05 | 178.00 | 177.31 | 177.10 | 177.07 | 177.36 | 177.75 |
| 16.000 | 359.71 | 358.46 | 180.75 | 178.83 | 177.90 | 177.57 | 177.50 | 177.86 | 178.34 |
| 18.000 | 1.00 | 359.62 | 181.50 | 179.39 | 177.92 | 177.24 | 176.93 | 177.07 | 177.45 |
| 20.000 | 3.33 | 1.93 | 183.11 | 180.40 | 177.32 | 175.00 | 172.84 | 169.35 | 167.20 |
| 22.000 | 23.04 | 23.07 | 198.80 | 192.06 | 170.49 | 135.71 | 103.41 | 81.58 | 76.97 |
| 22.400 | 44.00 | 45.66 | 215.65 | 204.67 | 165.56 | 128.98 | 111.40 | 101.87 | 100.01 |
| 22.500 | 50.44 | 52.80 | 220.48 | 207.86 | 165.26 | 132.48 | 117.95 | 110.22 | 108.77 |
| 23.000 | 62.79 | 70.97 | 222.03 | 204.05 | 170.29 | 155.61 | 149.76 | 147.33 | 147.16 |
| 26.000 | 6.05 | 3.84 | 184.92 | 180.08 | 171.10 | 168.30 | 171.98 | 177.90 | 179.65 |
| 30.000 | 7.07 | 3.06 | 186.32 | 177.81 | 171.74 | 35.48 | 108.65 | 181.45 | 186.43 |
| 33.000 | 11.86 | 6.11 | 190.95 | 170.21 | 22.01 | 20.54 | 25.68 | 185.31 | 189.56 |
| 36.000 | 33.26 | 24.22 | 212.11 | 79.33 | 38.50 | 37.84 | 39.42 | 204.56 | 210.90 |
| 37.000 | 65.59 | 54.64 | 244.36 | 90.81 | 69.96 | 69.39 | 70.59 | 235.77 | 243.17 |
| 37.400 | 88.83 | 76.95 | 267.56 | 110.02 | 92.90 | 92.35 | 93.44 | 258.54 | 266.39 |
| 37.500 | 92.13 | 82.93 | 273.85 | 115.39 | 99.13 | 98.59 | 99.65 | 264.62 | 272.67 |
| 38.000 | 122.43 | 108.68 | 301.10 | 139.29 | 126.09 | 125.59 | 126.51 | 291.13 | 299.92 |
| 40.000 | 161.03 | 134.61 | 339.50 | 170.89 | 163.56 | 163.11 | 163.71 | 324.86 | 338.29 |
| 45.000 | 174.47 | 14.26 | 352.31 | 178.78 | 174.95 | 174.46 | 174.64 | 224.83 | 350.79 |
| 50.000 | 178.04 | 5.94 | 354.99 | 180.16 | 176.71 | 175.94 | 175.74 | 184.50 | 352.55 |
| 60.000 | 182.65 | 6.35 | 355.78 | 182.15 | 175.32 | 170.64 | 165.13 | 168.54 | 335.83 |
| 70.000 | 192.65 | 15.11 | 336.30 | 189.15 | 102.89 | 38.11 | 30.51 | 32.91 | 206.74 |
| 76.000 | 242.12 | 64.16 | 268.98 | 235.17 | 79.32 | 73.36 | 72.22 | 74.32 | 250.97 |
| 76.800 | 264.52 | 86.52 | 287.22 | 257.17 | 99.89 | 95.02 | 94.14 | 96.22 | 273.04 |
| 76.900 | 267.58 | 89.57 | 289.89 | 257.17 | 102.76 | 98.01 | 97.15 | 99.23 | 276.06 |
| 77.000 | 270.59 | 92.58 | 292.40 | 260.58 | 102.76 | 98.01 | 97.15 | 99.23 | 276.06 |
| 80.000 | 326.32 | 148.17 | 340.27 | 263.50 | 105.55 | 100.93 | 100.10 | 102.17 | 333.93 |
| 90.000 | 348.94 | 170.53 | 355.28 | 264.42 | 175.12 | 175.44 | 176.32 | 178.57 | 356.73 |
| 100.000 | 340.36 | 163.81 | 345.74 | 184.38 | 175.59 | 179.16 | 181.29 | 184.04 | 2.76 |
| 110.000 | 99.28 | 99.28 | 279.86 | 125.37 | 159.18 | 209.96 | 218.98 | 223.83 | 44.16 |
| 110.800 | 290.01 | 112.37 | 292.99 | 133.32 | 158.78 | 219.52 | 229.76 | 235.20 | 56.01 |
| 110.900 | 291.83 | 114.13 | 294.76 | 134.55 | 158.78 | 219.52 | 231.08 | 236.61 | 57.49 |
| 111.000 | 293.64 | 115.89 | 296.53 | 135.81 | 158.94 | 220.45 | 232.36 | 237.97 | 58.93 |
| 120.000 | 354.67 | 175.22 | 355.25 | 180.59 | 169.20 | 181.34 | 190.37 | 195.78 | 16.33 |
| 130.000 | 3.15 | 2.01 | 185.38 | 180.59 | 132.35 | 167.86 | 183.94 | 189.55 | 9.04 |
| 140.000 | 11.00 | 7.90 | 191.00 | 191.00 | 36.54 | 82.73 | 186.29 | 194.15 | 13.67 |
| 150.000 | 30.50 | 30.66 | 214.47 | 214.47 | 48.36 | 55.12 | 205.01 | 218.00 | 37.65 |
| 154.000 | 76.85 | 255.91 | 69.45 | 86.39 | 86.39 | 90.75 | 240.64 | 257.83 | 77.54 |
| 154.700 | 87.28 | 266.32 | 79.58 | 264.23 | 96.48 | 100.57 | 249.99 | 268.18 | 87.90 |
| 154.800 | 88.79 | 267.83 | 81.04 | 265.72 | 97.94 | 101.99 | 251.33 | 269.68 | 89.40 |

| HEATZ | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 155.000 | 91.81 | 270.84 | 83.96 | 268.69 | 100.87 | 104.85 | 254.02 | 272.64 | 92.40 |
| 160.000 | 142.60 | 321.67 | 132.08 | 318.46 | 149.88 | 152.53 | 291.81 | 323.09 | 142.39 |
| 180.000 | 172.21 | 350.59 | 172.44 | 338.79 | 174.45 | 175.20 | 184.26 | 350.26 | 170.40 |
| 200.000 | 178.10 | 356.19 | 10.86 | 245.72 | 176.56 | 175.85 | 180.25 | 352.38 | 173.02 |
| 220.000 | 183.83 | 2.05 | 10.23 | 196.74 | 175.44 | 165.15 | 169.11 | 338.74 | 161.91 |
| 240.000 | 221.16 | 41.40 | 44.23 | 227.83 | 147.46 | 91.59 | 95.50 | 266.53 | 89.53 |
| 242.000 | 235.24 | 56.15 | 57.82 | 241.58 | 145.32 | 103.68 | 107.46 | 279.76 | 102.32 |
| 242.100 | 236.02 | 56.97 | 58.57 | 242.34 | 145.37 | 104.40 | 108.18 | 280.54 | 103.08 |
| 242.200 | 236.80 | 57.79 | 59.32 | 243.11 | 145.44 | 105.13 | 108.91 | 281.32 | 103.84 |
| 243.000 | 243.12 | 64.45 | 65.40 | 249.31 | 146.48 | 111.24 | 114.98 | 287.77 | 110.17 |
| 250.000 | 275.05 | 102.82 | 93.48 | 280.28 | 164.98 | 154.41 | 158.06 | 332.82 | 154.62 |
| 250.000 | 222.78 | 63.14 | 39.16 | 227.01 | 171.96 | 171.51 | 175.35 | 351.22 | 172.84 |
| 280.000 | 191.43 | 10.93 | 13.09 | 194.45 | 167.97 | 178.49 | 183.23 | 359.98 | 181.71 |
| 300.000 | 192.17 | 10.23 | 13.77 | 194.41 | 117.49 | 181.69 | 188.23 | 5.37 | 187.36 |
| 320.000 | 201.74 | 19.51 | 23.03 | 203.44 | 44.04 | 188.67 | 198.90 | 16.25 | 198.53 |
| 340.000 | 262.96 | 80.67 | 94.07 | 264.29 | 93.96 | 240.17 | 260.48 | 77.92 | 260.51 |
| 340.600 | 267.01 | 84.72 | 88.12 | 268.33 | 97.85 | 243.66 | 264.54 | 81.98 | 264.58 |
| 340.700 | 267.99 | 85.40 | 88.80 | 269.01 | 98.50 | 244.25 | 265.22 | 82.67 | 265.26 |
| 341.000 | 269.73 | 87.44 | 90.84 | 271.05 | 100.47 | 246.01 | 267.26 | 84.71 | 267.31 |
| 350.000 | 316.67 | 134.38 | 157.69 | 317.86 | 145.45 | 279.20 | 314.27 | 131.73 | 314.48 |
| 380.000 | 347.59 | 165.36 | 168.38 | 348.44 | 173.05 | 193.47 | 345.46 | 162.64 | 345.90 |
| 410.000 | 332.87 | 170.74 | 173.51 | 353.51 | 176.81 | 183.16 | 350.46 | 167.60 | 351.50 |
| 440.000 | 335.04 | 173.02 | 175.55 | 355.52 | 178.09 | 181.35 | 352.53 | 169.35 | 353.88 |
| 455.000 | 355.70 | 173.74 | 176.17 | 356.12 | 178.44 | 180.97 | 353.14 | 168.69 | 354.64 |
| 457.800 | 355.81 | 173.86 | 176.27 | 356.22 | 178.49 | 180.92 | 353.24 | 169.73 | 354.76 |
| 457.900 | 355.81 | 173.86 | 176.27 | 356.22 | 178.49 | 180.92 | 353.25 | 169.73 | 354.77 |
| 460.000 | 355.89 | 173.95 | 176.34 | 356.29 | 178.53 | 180.88 | 353.32 | 169.75 | 354.86 |
| 475.000 | 356.37 | 174.48 | 176.78 | 356.72 | 178.76 | 180.67 | 353.74 | 169.81 | 355.41 |
| 477.000 | 356.43 | 174.55 | 176.83 | 356.78 | 178.79 | 180.64 | 353.79 | 169.80 | 355.47 |
| 480.000 | 356.51 | 174.64 | 176.91 | 356.85 | 178.82 | 180.61 | 353.86 | 169.78 | 355.57 |
| 550.000 | 357.76 | 176.15 | 178.04 | 357.96 | 179.35 | 180.24 | 354.88 | 166.73 | 357.07 |
| 600.000 | 358.25 | 176.80 | 178.47 | 358.39 | 179.52 | 180.14 | 355.20 | 155.05 | 357.69 |
| 612.000 | 358.35 | 176.92 | 178.55 | 358.47 | 179.56 | 180.13 | 355.25 | 146.55 | 357.80 |
| 615.000 | 358.37 | 176.95 | 178.57 | 358.49 | 179.56 | 180.13 | 355.26 | 143.43 | 357.83 |
| 616.000 | 358.37 | 176.96 | 178.58 | 358.50 | 179.56 | 180.12 | 355.27 | 142.28 | 357.84 |
| 650.000 | 358.59 | 177.27 | 178.77 | 358.69 | 179.63 | 180.09 | 355.36 | 51.06 | 358.11 |