



MATH+METRIK ORPOPATED 274 SANDBANK ROAD P. O. BOX 220 CHESHIRE CONNECTICUT 06410

TEL. 272-3650, 272-9704

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Study of Dynamic Response in Stranded Wire Springs.

Final Technical Report

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Study of Dynamic Response of Stranded Wire Springs.

ABSTRACT

A total of 60 TD-recordings were made of 5 different test springs, 42 records in impact loading at velocities up to 560 inch/sec, and 18 records on the sudden release of spring, at extension velocities up to 830 inch/sec.

Each record was evaluated with respect to surge time (both wave head and wave peak), velocities, minimum height reached, etc...

For single wire springs, the formulas of the Surge Wave Theory were recapitulated and summarized in Section-C.In Section-D, corsecponding formulas were derived for stranded wire springs: These formulas use lumped values as parameters, such as the active weight of spring (or per coil) and the measured rate of the spring (or coil), instead of wine diameter d and mean coil diameter D_{o} ().

When using the <u>measured</u> initial rate of spring in these formulas, good agreement was reached between the calculated and the measured values. Therefore, a set of formulas is available now for stranded wire springs which predict dynamic values such as surge time, surge velocity, extension velocity in sudden release and induced load and stress within a few percent of the measured values.

Stranded wire springs were found to show a few phenomena over those observed on single wire springs: The surge velocity is not a constant for all compression levels of spring, but increases slightly with increasing compression. The slope of the wave head has a pronounced tendency to bracken during propagation, so that head and peak of wave have somewhat different surge velocities (about 10% difference). The wave head disappears in the records after from 2 to 3.5 surge times T, as compared to 6T or 8T in single wire springs.

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Study of Dynamic Response of Stranded Wire Springs.

A. Scope of Vork.

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1. Scope of work in general.

Photograph the transient wave propagation in stranded wire springs with a time-displacement camera under prescribed conditions of impact velocity, mass weight and spring preloads.

Evaluate time-displacement records and obtain measured values for time, coll displacement, velocity, acceleration and spring force. Determine the transient wave velocity, the dynamic spring force and stress from the TD-records.Some tests will also be performed on precompressed stranded wire springs released suddenly to determine expansion velocity and characteristics.

Conduct a theoretical analysis and establish formulae for the calculation of the dynamic load and stress and the transient wave velocity in stranded wire springs.

Correlate theoretical results with experimental data and prepare report on findings.

2. Number of test springs.

Five stranded wire test springs, two of the 3-strand type and three of the 7-strand type are to be investigated: 3-strand drive spring for SPIW Rifle; 7-strand drive spring for SPIW Rifle; 3-strand drive spring for Cal..50 M85 Machine Gun; 7-strand drive spring for Cal..50 M85 Machine Gun; 7-strand drive spring for Cal..50 M85 Machine Gun;

3. <u>Number of TD-records</u> to be taken.

The Contractor shall record a total of 50 time-displacement records, with usually 2 records each for a given condition of spring loading. These 24 pairs of records may be composed as follows:

5 types of stranded wire springs under impact, with two levels of preload each : 10 conditions 2 types of stranded wire springs under sudden release, with 2 levels of preload each : 4 conditions.

This makes for 14 different conditions or 28 records total, Another 22 records are extras, to be taken at different film speeds and magnifications, as the need arises during the program.

4. Evaluation of TD-records.

All the records will be evaluated regarding measured values of time, coil displacement, velocity, acceleration, surge velocity, spring force and stress. In addition, the <u>damping</u> of the surge waves over time and displacement will be evaluated. Also, it will be endeavored to measure the degree of damping of a particular spring by a numerical value, say by an exponent a in the energy term e^{-at}, supposed the energy decay follows such an exponential law.

5. Theoretical analysis of wave propagation in stranded wire springs.

The transient wavé vélocity will be derived from the General Wave Equation:

 $d^2y/dt^2 = c^2 \cdot d^2y/dx^2$. (1)

This general equation must be modified to account for the energy decay due to damping as inherent in stranded wire springs. At least an approximate solution will be sought for determining the influence of damping on the wave velocity.

6. <u>Technical Reports</u>.

1)

The results of the recordings and the analysis will be adaquately covered in the form of Technical Reports, 3 Quarterly Reports and one Final Report. The Third Quarterly Report may be included in the Final Report, if the Scope of Work has been completed by that time. These reports will contain some of the TD-records taken as examples, which are discussed in detail. Of the prints not contained in these reports, an extra set of prints will be forwarded separately.

B. Compiling basic spring date.

1. General.

On each of the 5 test springs, the basic dimensions were measured, before as well as after the testing. Some of the springs (SPIW-7 and M85-7) took a definite set during the testing, while the other 3 springs maintained their free height.

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The average pitch of the middle coils of the springs, h_f, at free height was also measured, over a length comprising from 30 to 60 coils. With h_f known, the displacement magnification q of a TD-record can be determined more accurately than from the free height h_f of spring, where the actual ends of the spring cannot be marked by special traces.

The force-deflection rate P/F of the springs was also calculated, by means of the conventional formula for single wire springs and simply adding the rates of the 3 or 7 single wire springs. Also, measured force-deflection diagrams P(F) were made available for each of the 5 test springs by AMSWE-RET. It was felt that a comparison between the calculated and measured spring rates (static rates) is a necessary part of this study.

2. SPIW drive spring. 3 strand.

The basic data are compiled in <u>data sheet X1</u>, in Appendix X. Its measured force inflection diagram P(F) is plotted in <u>Diagram 1</u>, Appendix X. This spring maintained its free height during the testing. The calculated rate P/F = 1.38 lbs/inch and the measured rate over the first 5" of stroke (1.40 lbs/inch) are in close agreement.

3. SPIV drive spring. 7 strand.

Here an inner guide rod of .266" was selected. The spring ends were found to have a somewhat lesser diameter than the middle coils, binding on a somewhat larger rod of .272" diameter.

Data sheet X2 in Appendix X gives the basic data and Diagram 1, page X3, illustrates the measured P/F-diagram.Here the rate P/F = 1.85 lbs/inch calculated under simplifying assumptions is substantially less than the rate P/F = 2.10 lbs/inch measured over the first 5" of stroke.This spring set from $H_{\rm F} = 14.26$ " to 13.80" during testing.

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4. 3-strand M85 drive spring.

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The <u>Busic data</u> are compiled in page X4 of <u>Appendix X</u>. This spring is rather husky, with an index D/d of only 8.62. Furthermore, it is coiled extremely stepp, with an uncorrected solid stress S_c approaching 190,000 psi.

<u>Diagram2</u> on page X6 of Appendix X illustrates the measured force-deflection diagram, P(R), according to the data furnished by AMSWE-RET. Over the first 5 inches of stroke, where the characteristics is rather linear, a spring rate R = 11.20 Ms/incn was measured. The spring rate calculated from the spring data [assuming 3 wires in parallel] is R = 10.45 lbs/inch, i.e., about 7 % less.

The spring experienced only a minor set during testing, from $H_{\rho} = 18.30$ " to $H_{\rho} = 18.20$ ".

5. <u>7-strand M85 drive spring</u>.

The <u>basic data</u> are compiled on page X5 of Appendix X and the measured force-deflection diagram P(F) is illustrated in <u>Diagram 2</u>, page X6, of the Appendix X. The calculated rate [parallel single wire springs assumed] of R = 3.29 lbs/inch was found to be substantially less than R = 4.0 lbs/inch as measured over the first 3 inches of stroke. Notice the great difference in rate and characteristics with respect to the 3-wire strand spring for the same gun.

This spring was found to take a substantial set during the testing, from $H_f = 18.26$ " to $H_f = 16.90$ ". This large set is mostly due to the omission of the buffer in a few tests, where the spring was then crushed solid by excessive kinetic energy of the impact mass.

6. 7-strand 20 mm gun drive spring.

The <u>basic data</u> are compiled on page X7 of the Appendix X and the measured force-deflection characteristics P(F) is illustrated in <u>Diagram 3</u>, page X8, of Appendix X. The calculated rate [parallel wires] of R = 6.25 lbs/inch is well below the measured rate of

R = 7.50 lbs/inch 9over the first 10 inches of stroke).

This spring took only a very minor set during the testing, from $H_f = 24.65$ " to $H_f = 24.5$ " and was found to have recovered to the initial free height after weeks. C. Summary of surge wave formulas on single vire springs. 1. Terminology. Dimensions, in [inch]. 8.0 Wire diameter đ Diameter of wire strand 3#-Outer diameter of coils **D**_ Inner diameter of coils D, Mean coil dismeter D Height or length of spring H Free height Hf Solid height H_ Initial height (preloaded) in impact loading H₁ Initial height in sudden release recordings H₂ Pitch of coil h Deflection of spring F Deflection per coul £

C1 .

3

x Wire coordinate, measured along the wire

L. Length of wire of one coil

L . Length of wire of entire opring (active coils)

y Deflection of a point of wire of coordinate x

b. <u>Times</u>, in[sec] or [ms].

t . Time in general (variable)

t = 0 Instant of impact or sudden release

T₁ Wave propagation time or "surge time", per coil

T Wave propagation time or "surge time" through spring (single length)

 T_h Surge time of the wavehead in first transition
T_s Separation time of rear spring end in sudden release
4T Period of free vibration of spring in sudden release, when rear spring end is fixed.

C1. Terminologyccontinued.

- c. Velacities, [inch/sec].
 - c Velocity of propagation of wave along the wive, also called "surge velocity".

- C2 -

v. Impact velocity of impact mass on spring,

also extension velocity of spring epa in sudden release $\mathbf{v}(\mathbf{x}, \mathbf{t})$ Velocity of any point on wire axis; of wire abscissa \mathbf{x} ,

,at the instant of time t.

▲ Kinetic amplitude of surge wave

d. Weights and forces, in [lbs].

w = m g Weight of spring (active coils only)

- $W = M \cdot g$ Weight of impact mass
- P Load at any spring point, also force at spring end
- P. Preload of spring in impact loading

P₂ Preload of spring in sudden release recording

- P_c Spring force at solid height
- P(F) Static force-deflection diagram
- △P Change of force induced by an impact of velocity v (Force amplitude of surge wave)

e. <u>Stresses</u>in [psi].

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G Modulus of clasticity in forsion

- S Stress at any point of spring (at wire surface), (shear).
- S, Initial (static) stress of preloaded spring (imp. t loading)

S. Static stress at solid height of spring

AS Change of stress induced by impact of velocity v (stress amplitude of surge wave)

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C1. Terminelogy.continued.

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- C4 -

2. Basic wave equation and surge velocity.

In the basic wave equation

$$d^2y/dt^2 = c^2 d^2y/dx^2$$
, (1)

the constant c denotes the velocity of propagation or"surge velocity" in the linear medium.For a helical compression syring, this surge velocity c is given by the formula

$$c^{2} = \frac{\frac{R_{Pf}}{\gamma/g} \cdot \frac{L_{1}}{(A + 4I_{p}/D^{2})}}{(A + 4I_{p}/D^{2})}$$
 [inch/sec]. (2)

where

R_{Pf} = R₁ : r'orce-deflection rate of one ccil , [lbs/inch/coil] A : Cross-sectional area of wire , [inch²] I_p : Polar moment of inertia of wire cross-section, [inch⁴]. For a round wire spring, these values are:

$$R_{Pf} = R_1 = \frac{1}{8} \cdot \frac{1}{D^3}$$
, (3)

$$I_p = (\tilde{V}/32) \cdot d^4$$
 (5)

The length of one coil, strictly speaking, is slightly longer than $\mathcal{T} \cdot \dot{D}$, by a factor k mlightly above 1:

$$L_1 = k \cdot \widetilde{T} \cdot D \qquad (6)$$

For a roundwire spring, the formula (2) for c then becomes;

$$c^{2} = \frac{d^{2}}{D^{2}} \cdot \frac{G \cdot g}{2 \gamma} \cdot \frac{k}{1 + d^{2}/2D^{2}}$$
 (7)

With k = 1.01 (conventional choice) and spring index D/t = 7 (mean value), the third factor in equation (7) becomes 1 and we have the simplified formula: $d = \sqrt{G + g^{T}}$

$$c = \frac{d}{D} \sqrt[3]{\frac{G \cdot g}{2\gamma}}$$
 (8)

For a spring from steel wire, this formula further simplifies to:

c = 88,560 . (d/D)

[inch/sec] 。

(9)

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3. Surge time and natural frequency.

Ve distinguish between the surge time T through the entire spring and the surge time T₁ per coil. In excluation of TD-records, the latter should be give, priority, due to the slight uncertainty regarding the accurate number of active coils in a spring:

$$T_1 = T/N = L_1/c = k \sqrt{\pi} \cdot D/c$$
 [sec/coil]. (10)

Assuming k = 1.01 as is conventional, it follows from (8) and (10):

$$\hat{T}_{1} = \hat{T}/N = \frac{3.17.D^{2}}{d} \sqrt{\frac{2.7}{g.G}}$$
 (11)

With mean values of spring steel:

$$T_1 = T/N = 35.8 \cdot 10^{-6} \cdot D^2/d$$
 (12)

When defining the active solid height

$$H_s = N \cdot d$$
, (13)

an alternate formula for the surge time is obtained:

$$T = 35.8 \cdot 10^{-6} \cdot (D/d)^2 \cdot H_s$$
 (14)

The latter form shows clearly, that the surge time T increases with the square of the spring index D/d.

Natural frequency:

If a spring is fastened with one end, the spring will oscillate at a fundamental natural frequency, where the period of vibration comprises 4 surge times T:

$$\dot{H}_1 = 1/4T$$
 [sec⁻¹] . (15)

With mean values of spring steel:

$$n_1 = 7000 \, d/(p^2 \, N)$$
 (16)

With both spring ends fastened, the period of the fundamental free vibration comprises only 2 surge times T and the fundamental natural frequency n_1 becomes twice the above value of equation (16).

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4. Single surge wave and amplitudes.

The forced motion v(t) of a spring and or of an intermediate point x of spring generates a "single surge wave", which has a kinetic amplitude of sume velocity v and a potential amplitude, which can be expressed in 4 different ways:

, in [1bs] As a change of load or force, Ap ; , in [psi] as a change of stress (uncorrected), ΔS ; as achange of deflection per coil. , in [inch/coil]; **Δ1** as a change of potential, , in [inch/sec]. ⊿u By definition, $\mathbf{u} = \mathbf{v}$.

The following formulas have beeb derived:

$$R_{1} = R_{1} \cdot T_{1} = (3.17/8) \cdot (d^{3}/b) \cdot \sqrt{2 \gamma} \cdot u/g^{2}$$
(17)

$$\Delta S/v = (\Delta P/v) \cdot (\Delta S/\Delta P) = \sqrt{2 \gamma \cdot G/g}$$
(18)

$$\Delta t/v = T_1 = 3.17 . (p^2/d) . \sqrt{(2\gamma/g.G)}$$
 (19)

$$\Delta u/v = 1$$
 (by definition) . (20)

With mean values for spring steel, these formulas specialize to:

∆p/v ≖	51.5 . d ³ /b	[lbs/inch/sec]	(21)
∆ 3/v =	131	[psi/inch/sec]	(22)
10/10	or o		

$$\Delta f/v = 35.8 \cdot 10^{-6} \cdot D^2/d$$
 [inch/coil per inch/sec].(23)

Here the surprising result is: The stress change induced in the spring by a forced velocity v (such as impact) is proportional to the velocity v, but independent from the spring dimensions d and D.Each inch/sec of forced velocity changes the stress of the spring point in question by 131 psi.

D. Modification of formulas for stranded wire springs.

- D1 -

1. Derivation of general formula for surge velocity.

We return to the basic wave equation (1) and to the general formula for the surge velocity, which is also voir for stranged wire springs:

$$e^{i2} = \frac{R_1 \circ L_1}{(\gamma'g) \circ (A + 4.1_p/D^2)}$$
, (2)

where $R_1 = R_{Pf}$; force-deflection rate of one coil, in [lbs/inch], and \tilde{U}_1 : length of wire in one coil, in [inch].

With the exception of the unusually husky 3-strand M85 drive spring (D/d = 8.62), the spring index D/d of the other 4 test springs ranges from 11.3 to 15. With an index of 14, the term $4.I_p/D^2$ drops to only 0.25% of the term A and therefore can be rightfully neglected. When further introducing the weight of one coil:

$$W_{1} = \gamma \cdot A \circ L_{1} \qquad [lbs/coil],$$

equation (2) becomes:
$$c^{2} = L_{1}^{2} \circ (R_{1} \cdot g/W_{1}) \qquad (24)$$

For stranded wire springs, this more general formula for the surge velocity should be used, for the following reasons:

- a) The length L_1 of one coil can be substantially longer than the conventional value of 3.17.D [i.e., k = 1.01 in formula (6)] for single wire springs.
- b) A stranded wire spring has a non-linear force-deflection characteristics, as the measured diagrams in Appendix X clearly show. The spring rate R of the entire spring and therefore also the spring rate R₁ of one coil increase with increasing deflection. This is due to the fact, that the wires are not only stressed in pure torsion as with a single wire spring, but have a more complex stress distribution pattern including tension, bending and also friction. There is no satisfactory and accurate method to derive the progressive forec-deflection characteristics of a stranded wire spring

to ob+ain a reliable formula.But measured values of the rates R and R₁ could be used instead.

For these reasons, it is preferred to leave formula (24) in its general form, with L_1 and R_1 as parameters.

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2. Derivation of formulas for surge times.

From equation (10), the surge time per coil, T_1 , and of the entire spring follow to be:

D?-

$$L_1 = L_1/c$$
 [sec] , (10a)

and

$$T = N_0 T_1 = N_0 L_1 / c = L / c . [sec] (10b)$$

When introducing c from formula (24), it follows:

Т

$$T_1 = \sqrt{W_1/(R_1 \cdot g)}$$
 [sec] , (25)

[se/:

(26)

and

Here W denotes the active weight of the entire sping and may be calculated from the formula:

W/R.g

$$I = N.W_1 = 0.785 d^2.L_1$$
 [3bs] . (27)

Here the wire length per coil, L_1 , has to be calculated and for the rate $R_1 = N.R$, the measured value of R may be used.

The surge times T_1 and T are proportional to the square root of the weight per coil, W_1 , and inversely proportional to the spring rate per coil, R_1 . The surge time T,of course, is also proportionate to the number of active coils,N.

Natural frequency:

For a spring with one fixed end and one free end, the period of free vibration comprises again 4 surge times and the fundamental natural frequency becomms:

$$n_1 = 1/4.T$$
 [sec⁻¹]. (15)

And for a spring with both ends fixed, the period of free vibration comprises only 2 surge times and the fundamental natural frequency becomes:

$$n_1 = 1/2T$$
 [sec⁻¹]. (15a)

3. Amplitudes of single surge wave in stranded wire springs.

- D3 -

The impact of a mass on a spring end with velocity v (or any forced motion) generates a "single surge wave", which has a <u>kinetic amplitude</u> (coils are forced to move with same velocity v) and a <u>potential amplitude</u>, which can be expressed in various ways: As a change of deflection per coil, Δf , in [inch/coil]; as a change of load or force, ΔP , in [lbs]; as a change of stress (uncorrected), ΔS , in [psi].

Deflection per coil:

During a time interval equal to the surge time per coil, T_1 , the surge waves traverses one coil, while the end of coil makes a displacement (v.T₁). Therefore, the deflection per coil becomes:

$$\Delta f = T_1 \cdot v$$
 [inch/coil]. (28)

Load induced:

This deflection per coil induces an additional force

$$\Delta P = R_{i_1} \cdot \Delta f = R_{1} \cdot T_{1} \cdot v \qquad [1bs] \qquad (29)$$

Stress induced:

In a <u>single wire spring</u>, the maximum torsion stress S (uncorrected) has been found to be independent from the dimensions d and D of the spring:

$$\Delta S = \sqrt{2\gamma \cdot G/g} \cdot v \qquad [psi] , (18)$$

or, for a steel spring: $\Delta 8 = 131 \cdot v$ [psi]. (22)

In a <u>stranded wire spring</u>, however, the stress distribution pattern is more complex. The impact energy is not only stored in torsion stresses, but also in bending and tension stresses (besides friction), Therefore, the actual maximum torsion stress will be <u>slightly less</u> than the value calculated from above formulas (18) and (22).

4. Calculation of wire length per coil.

The wire length per coil, L_1 , enters into the formula for the surge velocity c and also into the formulas for the surge times, by way of the weight per coil, W_1 . Therefore, L_1 must be calculated as accurately as possible.

- D4 -

The coll length L_1 of a single wire in a stranded wire spring is greater than the projected circumference 3.14.D, for 2 reasons:

- a) due to the pitch of spring at free height, h_{t} ;
- b) because the wire (except the core wire) forms a spiral or -

helix around the centerline of the straight or coiled strand. <u>Table I</u> on page Y1 of <u>Appendix Y</u> illustrates the procedure of calculating L₁ for the 5 test springs under consideration.

The length of the coil centerline follows from the formula:

$$L_1' = \sqrt{(3.14.D)^2 + h_f^2}$$
 [inch]. (30)

The ratio L₁'/3.14.D is found to be from 1.018 to 1.034, ises, a percentage increase of the centerline length from 1.8% to 3.4% over 3.14.D.

The second step is the calculation of the <u>length of the spiral of</u> <u>a wire</u> with respect to the centerline of the strand: First, the number of wires wrapped around the centerline over one coil was measured on the test springs, ranging from 7 turns for the M85-3 spring to 17 turns for the 20-mm-7 spring.From this, the length of one lay, designated by b , was calculated.b was found to range from .367" to 1.15".

Second, the radius a of the spiral was determined (see illustration of radius a on the bottom of Table I), as well as the circumference $2\sqrt[7]{a_0}$. The increase of the wire length L_1 over that of the strand centerline is then described by the ratio:

$$L_1/L_1' = b^*/b = \sqrt{1 + (2\pi a/b)^2}$$
. (31)

This length increase was found to be from 4.5 to 7.6 %, i.e., about twice as as great as the relative length increase from the pitch.

Altogether (pitch and spiral effect combined), the length L_1 of a single wire coil is <u>substantially greater</u> than 3.14.D, by from 7% (SPIW-3 spring) to 10.8% (20 mm-7 spring).

5. Numerical calculation of surge times and surge velocity.

- D5 -

The results of these numerical calculations are given in <u>Table II</u>, page Y2, in Appendix Y.

First the cross-sectional area of the individual wires were calculated and added up to the cross-sectional area A of the entire strand. By multiplication with the length of coil L_1 (from Table I), the weight per coil, W_1 , is obtained. The active weight of the spring has also been calculated in Table II.

We now consider the formulas (25) and (26) for the calculation of the surge times T_1 and T.Due to the non-linear force-deflection characteristics (see Diagrams 1 to 3 in Appendix X), it becomes here somewhat doubtful, what values should be used for the spring rates R_1 and R.Since no decision can be made prior to a comparison with the test results, two alternate calculations have been made using the following assumptions:

Assumption A:

Here the spring rate of each single wire in the strand was calculated by means of the well-known formula for single wire springs and these rates were simply added to obtain the rate of the stranded wire spring. These rates were already given in the <u>data sheets</u> of the springs in Appendix X.

Assumption B:

Here the rate of the spring was evaluated from the measured force-deflection diagrams of the springs (<u>Diagrams 1 to 3</u> in Appendix X). Only the initial linear portion of the diagram was used, over a stroke anywhere from 3" to 10". These rates were also given in the data sheets of the springs in Appendix X. The measured rates were found to be always higher than the satculated rates, with deviations up to 20%. The deviations were found to be greater with 7-wire strand springs than with the 3-wire strand springs.

Under each assumption, the surge velocity c has also been calculated for the 5 test springs. With assumption B, shorter surge times and higher surge velocities are obtained.

6. Numerical calculation of amplitudes of single surge wave.

- D6 -

In a single surge wave, the potential amplitude is always propose tional to the impact velocity v.It can be expressed either as deflection per coil, AT induced load, AP induced stress, AS .

Deflection per coil, Af :

According to formula (28), the deflection per coil Δf generated by 1 inch/sec of impact velocity is equal to the surge time per coil, T_1 Therefore, the T_1 -values given in <u>Table II</u> of Appendix Y already establish the ratio $\Delta f/v$ for each of the 5 test springs. For example (using assumption B), in the SPIW-3 spring, each inch/sec of impact velocity generates adeflection per coil $\Delta f = .000157$ [inch]. With a total deflection per coil $f_s = h_f - d^* = .231 - .073 = .158"$, it would take an impact velocity $v_0 = 1000$ inch/sec $D^*_{1,2}$.

The <u>reverse holds also true</u>: A precompression by .000157 inch/coil will generate an extension velocity of 1 inch/sec, upon sudden release. If the spring is suddenly released from solid height, an extension velocity $v_0 = 1000$ inch/sec can be expected. This is the <u>limit velocity</u> of the spring.

Induced force, $\triangle P$:

According to formula (29), the ratio $\Delta P/v$ is equal to $(R_1 \cdot T_1)$ and these numerical values are also given in Table II.For example (second assumption), in the SPIW-3 spring, each inch/sec of impact velocity induces an additional force $\Delta P = .0163$ [1b]. Or, in other words, an impact velocity of 1/.0163 = 61.5 inch/sec is required to raise the force of the spring by 1 lb.

Induced stress, ΔS :

Here it is suggested to use the formula for single wire springs,(18), which takes the very simple form of formula (22) for steel springs. Each inch/sec of impact velocity raises the stress by 131 psi . This relationship is considered a good approximation also for stranded wire springs, where the force-stress relationships are more complex than in a single wire spring.

E. Preparation of test fixtures for recording spring response.

÷ E1 -

1. Fixture for impact-loading of springs.

On a heavy and rigid bench stand,2 end supports were mounted at a distance of about 56", to receive the ends of an inner guide rod. This guide rod consists of two sections:

A righthand section of .812" diameter guiding the accelerating spring, which can be preloaded to various heights to impert any desired velocity to the impact mass.

A lefthand section of rod to guide the test spring. This section was made interchangeable, having diameters of .266", .300", .375" and .750".

Between the accelerating spring and the test spring, the impact mass is arranged. This mass is guided on the outside, in a tube of 1.50" inner diameter. This was made necessary by the rather thin guide rods for some of the test springs, which were much too flexible to guide a heavy impact mass. The guidetube contains a longitudinal window for viewing of the test spring by the time-displacement camera and also longitudinal slots for a cocking handle of the impact mass and for triggering means.

The opening of the camera shutter is synchronized with the release of the impact mass from a sear mechanism.

An impact mass of about 2 lbs had been specified.However, the 5 test springs were found to be widely different in energy capacity and it was found proper to adjust the weight of the impact mass somewhat to the strength of the springs.Therefore, 3 different weights were used in this test program:

SPIW test springs	:	W	*	1.15	lbs	,	L	=	3.15"	;
<u>M85 test spring</u>	:	W	æ	2.15	lbs	,	L	×	6°05"	ن •
20 mm test spring	:	W	×	3.25	lbs	,	L	×	5.25"	。 ∽

The specified range of impact velocities

v_o from 30 to 40 ft/sec was somewhat amplified to range from

 $v_0 = 25$ ft/sec to close to 50 ft/sec.

2. Fixture for sudden release of preloaded springs.

- E2 -

Hère thé 2 end supports were mounted closer together, at a distance of about 34^{16} , receiving the ends of the inner guide rods of the test springs.3 of the 5 test springs were submitted to sudden release tests: SPIW-3, SPIW-7, and M85-7 drive springs.

A sear mechanism had to be developed for the sudden release of the precompressed springs. A sear blade holds one side of the end coil of spring, also touching the guide rod. Then this sear blade is suddenly withdrawn by the core of a solenoid, in synchronisation with the opening of the camera shutter.

The test spring was always released alone, with no adjacent mass +: be driven. Two levels of preload were used, one close to solid compression and the other close to half compression. Here the rear end of spring was only supported in the rearward direction, so that it could fly off forward at the end of the spring expansion.

In another arrangement, the rear spring end was fastened to the inner guide rod, permitting no freedom in either direction. In this case, the spring performs a free vibration, in periodic fashion.

P. <u>3-strand SPIW drive spring</u>: <u>Discussion of TD-records on impact-loading</u>.

- F1 -

1. <u>Review of TD-records delivered</u>.

A total of 8 TD-records was taken and delivered: Two each for each of the 4 combinations of high or low impact velocity v_0 and no prelead (spring at free height H_p) and a substantial preload. 4 of these TD-records are contained in this report, while a second set of 4 TD-records taken under the same operating conditions is being delivered separately.

Operating conditio	n,	No. of record	No. of record in second set	
Impact velocity	Precompression	in report		
high	none	70-723.	722	
high	yes	725	726	
low	none	728	727	
low	yes	730	731	

All records on photo-recording paper.

<u>Note</u>: A block of 20 consecutive numbers has been reserved for the TD-records taken on the 3-strand SPIW drive spring. On impact-loading: TDR 70-721 to 70-732. on sudden release : TDR 70-733 to 70-740 (See Section G).

2. Discussion of TD-record 70-723. (high, no preload)

Operating conditions:

The timing marks given at intervals of 1 millisec indicate a film speed p = 201 inch/sec , with the time axis running from right to left. The magnification

q = .387x

has been determined from the distance of the horizontal coil traces prior to impact, when the coils were at rest and at free height. A rubber buffer has been installed in parallel to the test spring, to be contacted at a height of test spring H = 7". The impact velocity of the impact mass of 1.15 lbs was evaluated to be

 $v_0 = 478$ inch/sec.

Response of spring:

The test spring decelerates the motion of the impact mass only to a minor extent, with the additional buffer doing most of the stopping. The minimum height $H_{min} = 5.50^{"}$ observed comes very close to the solid height H_{g} of the spring. The impact mass leaves the spring again at a rebound velocity $v_{0}^{"} = 294$ inch/sec.

- F2 -

The impact of the mass generates a step wave, which traverses the spring at evidently constant surge velocity. However, while the acceleration of the front coils is rather sharp and almost instant-like, the acceleration of the later coils comprises a longer and longer time interval, as the wave progresses. For this reason, it becomes difficult to determine the surge time and surge time per coil accurately. In reading the surge time per coil,

 $T_1 = .160 \text{ millisecond/coil}$

the very first motion response of the previously resting coils has been considered. The surge time T, however, was read in such a way, that the wave had reached a spring end with its full amplitude rather than with its faint head only. It was found:

T = 11.85 milliseconds.

Up to the time t = 2T, with only 2 waves superposing, the imposed deflections per coil are absorbed without clashing between coils. After t = 2T, however, due to the superposition of 3 wave amplitudes, the front coils start to clash and this clashing is propagated toward the rear coils in a much shorter time interval than T.Notice the almost vertical wave front of the wavehead after 3 superpositions.

After about 3 transitions of the wave, in spite of the high impact velocity and steep wavehead, the wavehead cannot be discerned anymore. The coils now share rather equally in the overall deflection imposed on the spring, which is a most desirable feature. This quick equalization of the deflection over all the coils is much more pronounced than in a single wire spring and is evidently caused by high inherent damping.

3. Discussion of Therecord 70-725. (high with preload)

- F3 -

Operating conditions:

Here the test spring was precompressed, to a height $H_1 = 13.0$ ". The pitch h_1 of the precompressed coils had been measured prior to recording and the magnification q = .387x was confirmed by this measurement. The impact velocity of the mass is found from the record:

 $v_0 = 465$ inch/sec.

Response of spring:

Again, the test spring fails to decelerate the impact mass substantially and reaches again a minimum height of 5.5", close to solid height. The front coils are somewhat hidden by the precompressing means]. The surge time T to the rear end of spring is measured to be T = 12.0 ms

and the surge time per coil is found to be $T_1 = .155 \text{ ms/coil}$. After t = T, with the start of 2 wave amplitudes being superposed to the precompression, the rear coils clash and the reflection of the wave to the front end of spring takes only about half the surge time T, due to the continuous clashing of coils. This must be considered a rather severe case of impact loading. Nevertheless, during the expansion phase of the spring, the originally very pronounced wave head can no longer be distinguished and the coils share rather equally in the total deflection.

4. <u>Discussion of TD-record 70-728</u> (low without preload). <u>Operating conditions</u>:

Here the impact velocity of the mass has been greatly reduced to $v_0 = 317$ inch/sec .

Response of spring:

Here the spring is more efficient in decelerating the slower mass and does not come quite so close to solid height:

$$H_{\min} = 5.95$$
 inches.

The surge times are found to be about the same as with the higher velocity level: T = 12.05 ms

and $T_1 = .160 \text{ ms/coil}$

When the surge wave reaches the front end of spring again t = 2T, the superposition of 3 waves is fully absorbed my the spring, without clashing of coils. After t = 3T, the tear coils appear to become solid, with very minor clashing. The wavehead can be distinguished up to t = 4T, though quite faintly only.

5. Discussion of TD-record 70-730 (low with preload).

--- PA ---

Here the spring is again preloaded to a height of about 13^n and is struck by the impact mass at a velocity

 $v_0 = 322$ inch/sec.

Response of spring:

The spring reaches a minimum height $H_{min} = 6.15"$, above solid height. Up to t = 2T, i.e. 2 wave superpositions, the coils do not clash and the wavehead evidently moves at constant velocity along the spring wire.After t = 2T, when the wavehead reaches again the front end of spring, 3 waves superpose to the initial deflection f_1 due to preload and this deflection cannot be absorbed anymore within the available clearance between coils.Rather, the front coils clash violently and this clashing is propagated toward the rear end of spring at much greater speed than within the first 2T sec.Notice the almost vertical front of the wavehead after t = 2T.However, after t = 3T, the wavehead cannot be discerned anymore. G. <u>3-strand_SPIW_drive_spring</u>: Discussion of TD-records on sudden release of spring.

---- G1. ---

1. <u>Review of TD-records delivered</u>.

A total of 6 TD-records was taken and delivered as follows:

Operating c	ondition	No. of record	No. of record		
Preload Support of rear end		in report	in secont set		
high	rearward only	70-734*	733*		
medium	rearward only	735*	736*		
medium	both ways (fastened)	738*	737*		

The asterisk * denotes that these records were recorded on film rather than on paper. Therefore, no explanatory lines are shown in the prints, since they are not feasible on the film.

2. Discussion of TD-record 70-734*. (high)

Operating conditions:

Here the spring was precompressed to a height $H_2 = 7$ inches, i.e., not far from the solid height H_s of about 5.5". The pitch of the 60 middle coils was measured on the spring to be L = 15.64", or a pitch $h_2 = .094$ inch. This establishes the displacement magnification q = .395x.

Response of spring:

Upon the sudden release of the front end of spring, the latter expands with a practically constant velocity, which is measured to be:

$$v_0 = \frac{832}{inch/sec}$$

Coil after coil follows in the release and the rear end of spring lifts off its rear one-way support after a time

By theory, this separation time T_s is equal to the surge time T. The rear end also reaches a maximum expansion velocity of about 850 inch/sec, but its velocity oscillates slightly about a mean value of roughly 830 inch/sec.Again, the originally step wave head broadens to a more and more wider and lesser slope, as seen in the record. This smoothening of the wavehead also accounts for the disturbance superposed to the expansion pattern of the spring coils at constant velocity.

3. Discussion of TD-record 70-735* (medium).

Here the spring is precompressed to a height of about .2 inches. More accurately, the 60 middle coils have an average pitch at precompression $h_2 = .161$ inch/coil .

Response of spring:

The front expands suddenly with a rather constant velocity $v_0 = 416$ inch/sec The release is propagated through the spring evidently at constant velocity and the rear end of spring lifts off its support after

 $T_s = 12.4 \text{ ms}.$

- G2 -

4. Discussion of TD-record 70-738* (medium, rear end fixed).

Here the rear end of spring is fastened to the inner guide rod, rendering about one coil inactive. The spring then was precompressed to about the same height of 12 inches.

Response of spring:

The front end of spring performs a continuous vibration, extending at constant velocity over 2 surge times and then compressing over 2 surge times, etc.. The entire spring performs a vibration with the period

indicating a surge time. T = 12.2 ms.

At multiples of the surge time after spring release, the spring coils have the same potential and also the same velocity, in the following pattern:

Instant of time			of tim	0	Velocity		Pótential		
t	۲,	0				0		iŋ	compression
t	×	2'			in	extens	ion		0
t	**	2Ť	•			0		in	extension
t	XX	3Т			in	pompre	ssion		0
t	×	4T				o •		in	compression
an	d f	this	cycle	repeats	itself, with so	mewhat	decrea	sine	amplitude.

₩ H1/ -

н. <u>7-</u>

7-wire SPIW drive spring:

Discussion of TD-records on impact loading.

1. <u>Review of TD-records taken</u>.

Time-displacement records on the 7-wire SPIW spring are numbered from 70-741 to 70-760. The records on impact loading number from 741 to 750 as follows:

Operating condition	ons ·	No. of record	no. of record in second set	
Impact velocity	Precompression	in report		
high	nçne	744	743	
high	yes	746	745	
low	none	748	747	
low	yes	749	750	

All these records on photorecording paper.

Note : It was found rather difficult to obtain strong enough traces over the entire length of spring, especially for the end coils. This is chiefly due to the configuration of the strand: The individual wires are rather thin and also give a favorable light reflection over a small longitudinal section only. Therefore, a stranded wire spring must be expected to give poorer TD-traces than a single wire spring of the some size. This deficiency has led to have some of the records recorded on very fast film rather than paper, e.g., all the records on sudden release of spring and also some on impact loading.

2. <u>Discussion of TD-record 70-744</u> (high, no preload). <u>Operating conditions</u>:

Here the spring had already set to a free height $H_f = 13.80$ " (from 14.26"), due to extensive preliminary testing. The free pitch h_f of the 50 middle coils had been reduced to .231", from .238". As with the 3-strand SPIW drive spring, an impact mass of 1.15 lbs was used and the additional rubber stop was contacted by the mass at the same height H = 7". The impact velocity was measured to be :

 $v_o = 480$ inch/sec.

- H2 -

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Response of spring:

The spring is being compressed to $H_{min} = 5.6$ ",i.e.,close to the solid height of about 5.2". The main deceleration is provided by the additional rubber buffer, between the heights H = 7" and 5.6".

The acceleration of the spring coils, which is instant-like for the front coil, spreads over a longer time interval, as the transition of wave progresses. The time T_h is defined as the transition time of the <u>head of the wave</u>, as far as is can be discerned in the record:

 $T_{\rm in} = 9.0$ ms. Likewise, the surge time per coil, T_1 , is determined using the head of the wave: $T_1 = .158$ ms/coil. On the other hand, the surge time T is measured to the point where the wive has reached a spring end with its <u>full amplitude</u> and therefore is found to be somewhat larger than $T_{\rm b}$:

3. Discussion of TD-record 70-746 (high, with preload).

Here the spring is precompressed at a height $H_1 = 11"$ (from 13.8") and is struck by the impact mass at a velocity $v_0 = 485"/s$. Response of spring:

Again, the spring is being compressed to $H_{min} = 5.6$ ", close to: its solid height, with the rubber buffer doing the essential stopping of the impact mass.

During the first transition of wave, from the front to the rear, a surge time T = 9.0 ms is measured. The head of the wave (only faintly discernible), however, travels somewhat faster, requiring a transition time

 $T_h = 7.5$ ms only. Likewise, the surge time per coil T_1 (based on the progression of the wavehead) is found to be quite less than with the free spring:

$$T_1 = .139 \text{ ms/coil}$$
.

Therefore, it appears that the preload promotes the smoothening out of the wavehead and raises its surgevelocity.

After t = T, the rear coils start to clash. Evidently, the superposition of 2 waves to the preload exceeds the clearance between coils.

4. Discussion of TD-record 70-748 (low, no preload).

Here the spring is struck by the impact mass at its free height $H_{p} = 13.8^{"}$ with a velocity $v_{0} = 315$ inch/sec.

- H3 -

Response of spring:

The spring reaches a minimum height $H_{min} = 6^{m}$, in the range of the additional rubber buffer, which still performs the major work in stopping the impact mass.

From the first transition of wave, the following surge times were read: T = 9.85 ms; $T_q = .163$ ms/coil.

Up to the time t = 3T, the wave progresses through the spring without the clashing of coils and the surge times for the first, second and third passage through the spring are practically the same. After t = 3T, the rear coils show minor clashing; therefore, the superposition of 4 waves cannot be absorbed anymore by the original clearance between coils. However, all coils shore rather equally in the total deflection of spring and no definite wavehead can be found after t = 3T.

5. Discussion of TD-record 70-749 (low, with preload).

Here the spring i. again preloaded to a height $H_1 = 14^{\circ}$ (from 13.8") and is struck by the impact mass of 1.15 lbs at a velocity

v_o ≖ 335 "/s .

Response of spring:

Here the end coils are partly hidden, due to some change in the position of the illuminating lamp, for stronger traces of the more important front coils.

The following surge times are read from the first transition:

T = = 9.4 ms; $T_1 = .143 \text{ ms/coil}$

These times appear to be slightly shorter than without preload of Spring (TDR 70-743 above).

The second transition of wavehead takes about the same time T, without clashing of coils. However, after t = 2T, the front coils come close to clashing. But no wavehead can be discerned after t = 2T. Ś

I. <u>7-wire SPIW drive spring</u>:

Discussion of TD-records on sudden release of spring.

- I1 -

1. <u>Review of TD-records taken</u>.

A total of 6 TD-records was taken and delivered as follows:

Operating a	conditions	No. of record	No. of record in second set	
Preload	Support of rear end	in report		
high	rearward only	70-751*	753*	
medium	rearward only	755*	75.6 *	
medium	fixed both ways	759*	760*	

All records taken an fast film. Thereforé, no explanatory lines could be drawn on the original.

2. <u>Discussion of TD-record 70-751*</u> (high, one-way support). <u>Operating conditions</u>:

These tests were taken before the impact tests, with the spring still at its original free height, $H_f = 14.26$ ". From the 50 middle coils, the average pitch at free height was measured to be: $h_f = .238$ ". The spring was then precompressed to a height $H_1 = 6.5$ ", i.e., an average pitch of the 50 middle coils $h_2 = .107$ ". This makes for a deflection per coil $f_2 = .131$ ". Notice the higher film speed p = 392 "/s .

Response of spring:

Upon the sudden release of the front end of spring, the latter expands at a constant velocity

v_o = 811 "/s . Coil after coil follows in the release and the rear end of spring lifts off from its one-way support at a time

$$T_s = 9.9 \text{ ms}$$

after release, which theoretically equals the surge time T. The head of the release wave (extension wave), however, traverses somewhat faster through the spring, requiring only a time

 $T_h = 7.8 \text{ ms}$. (Notice third coil from rear).

3. Discussion of TD-record 70-755* (medium, one-way support).

- IŹ -

Here the spring is procompressed to a height $H_2 = 10"$, or, more accurately, to an average pitch $h_2 = .167"$. This means a deflection per coil $f_2 = .238 - .167 = .071"$.

Response of spring:

Here the camera shutter closed rather early; but still the most important time phase including the lift-off of the spring from the rear support was covered.

Extension velocity of front end of spring:

 $v_0 = 432$ inch/sec.

Separation time of rear end from support:

$$T = 10.2 \text{ ms}$$

Propagation time of wavehead through spring (see third coil from rear): $T_h = 9.0 \text{ ms}$.

Here the two times T_s and T_h are relatively closer together than with the previous case of spring release from high preload.

4. Discussion of TD-record 70-759* (medium, rear end fixed).

Here the rear end of spring was fastened to the inner guide rod, rendering about one coil inactive. The spring was precompressed to an initial height H_2 of about 10".

Response of spring:

The entire spring performs a periodic vibration with the period

		4T =	40.8	ms,		
or	à surge time	T =	10.2	ms .		and the second se
	The spring coils	underg	o alterr	nating phases	of	
	no st	ress	,	velocity in	extension	
	extension	stress	8	velocit	y 0	
	no st	ress	,	velocity in	compression	
	compressio	n stres	s,	velocit	y 0	, etc
-	.					

The front end of spring extends and compresses at constant velocity, with its TD-trace being of saw-tooth shape. This is definitely not a harmonic or sinusoidal vibration of spring, but a special mode including many higher harmonics (see Fourier series).

J. <u>3-strand M85 drive spring</u>: <u>Discussion of TD-records on impact loading</u>.

1. <u>Review of TD-records taken</u>.

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Time-displacement records on the 3-wire M85 drive spring are numbered from 70-761 to 70-780, with the following records being delivered:

- J1 -

Operating conditi	ons	No. of record	No. of record	
Impact velocity	Precompression	in report	in second set	
high.	none	• 764*	763	
high	yes:	767*	766	
low	none	769	768	
low	yes	772*	770	

Note : A record number with * denotes a record on file rather than on photo-recording paper.

2. <u>Discussion of TD-record 70-764*</u> (high, no preload). With an impact velocity

 $v_0 = 473$ inch/sec

the spring reaches a minimum height $H_{min} = 7.45$ ", which is only 0.30" less than the height at buffer contact. Considering the highly nonlinear force-deflection characteristics of the additional rubber buffer, it can be said that the impact is being stopped practically by the spring alone.

The surge time is measured to be T = 7.95 ms. The coils of the spring come very close to clashing after t = 3Tand t = 4T (4 single surge waves superposed).

It should be stated here that this spring is coiled very steeply, having an uncorrected solid stress of more than **185**,000 psi,or a velocity potential in excess of

 $u_g = 120 \text{ ft/sec or } 1440 \text{ inch/sec}$

3. Discussion of TD-record 70-767* (high, with prelead).

≚ J2 -

In spite of the preload, the spring is found to stop the impact of the mass practically on its own; without the help of the additional buffer $(H_{min} = 7.7"$ near buffer contact).

The spring coils come close to solid after t = 2T and contact each other after t = 3T (see rear coils). The wavehead is visible over a time phase up to t = 3.5.T. From there ongall the coils share about equally in the total deflection of the spring.

The surge time is read to be T = 7.9 ms.

4. Discussion of TD-record 70-762 (low, no preload).

With the lower impact velocity

v_o = 382 inch/sec

the spring is effective to stop the impact of the mass at a minimum height $H_{min} = 9.35"$,

i.e., about 2" above solid height.

The coils do not clash and the wavehead is visible up to t = 4T. The surge time is measured to be

≖ **7.80** ms.

5. Discussion of TD-record 70-772* (low, with preload).

With an impact velocity

Ť

 $v_0 = 420$ inch/sec, plus preload, the impact of the mass is stopped at a height

 $\frac{H_{min}}{hout} = 9.2"$ about 2" above solid height.

With preload, the wavehead disappears somewhat sconer, after $t = 3T_{\circ}$ Also, the surge time

r = 7.7 ms

is measured to be somewhat shorter than without preload.

Note : It should be understood, that the impact velocity v of the mass could not be evaluated to highest accuracy standards, due to the shortness of the available trace. Also, the recording edge of the mass is in a different optical plane than the coils. Therefore, the actual impact velocity may be 1 or 2 % less than given.

K. <u>7-strand M85 drive spring</u>: <u>Discussion of TD-records on impact loading</u>.

1. <u>Review_of TD-records taken</u>.

The TD-records on impact loading are numbered from 70-781 to 70-792, with the following records being delivered:

- K1 -

56t

2. Discussion of TD-record 70-782 (no preload, no buffer).

In this recording as well as in a few previous loadings, the impact mass failed to engage the buffer, so that the spring had to absorb the energy of impact alone. As a result, the spring took a substantial set from its original free height $H_f = 18.25$ ". Prior to this recording, the spring had a free height $H_f = 17.25$ ".

With an impact velocity of $v_0 = 490$ inch/sec, the spring is being crushed solid to

 $H_{\min} = 5_0 4'',$

which is somewhat less than the static solid height of about 6". Notice the minor reduction of compression velocity over the spring stroke.Only in the last 1%2 inch of spring stroke near solid height, the essential deceleration of the impact mass takes place.In other ree cordings with buffer engaged, the deceleration phase extends over more than 1 inch.

After $t = 2T_{g}$ when 3 amplitudes of surge wave superpose, the front coils start to clash.Notice the much greater velocity of propagation of the wave head from then on, which is about 3 times the normal propagation velocity, with clearance between coils.

The surge time through the spring was measured to be

T = 10.8 ms.

3. <u>Discussion of TD-record 70-783*</u> (high, no preload, but with buffer). Hore the free height had been further reduced by setting, to 10.9". With an impact velocity v₀ = 481 inch/sec , the spring comes down to H_{min} = 0.1" ; which is about the solid height of the spring in static testing. The main deceleration of the impact mass is performed by the additional buffer, which is compressed over 1.65" stroke, from H = 7.75" to H_{min} = 6.1". Notice the more gradual deceleration of the spring end in this record as compared to TD-record 70-782 without buffer.

Again, the coils start to clash after t = 2T, when the amplitudes of 3 waves superpose.

The surgo time per coil of the wave head (first transition) was measured to be

 $T_1 = .235 \text{ ms/coil}$

i.e., about the same as in TD-record 70-782.

The entire surge time through the spring is found to be: T = 11.7 ms.

4. Discussion of TD-record 70-786* (high, with preload).

Here the spring was precompressed from a free height $H_f = 16.90^{\circ}$ to an initial height $H_f = 14.5^{\circ}$.

With an impact velocity $v_0 = 474$ inch/sec, the spring coils at the rear end go practically into contact after t = T, but without clashing. The time for the second transition of the wave head is slightly less than the time for the first transition, indicating coil contact here and there.

The wavehead returns to the front end of spring near the end of the compression stocke and then the front coils clash, as indicated by the much steeper wave front, which is visible up to about t = 2.5.T. Again, the spring reaches practically solid height, with $H_{min} = 6.05$ ".

During the first transition, the surge time per coil of the wave head is measured to be $T_1 = .209 \text{ ms/coil}$. This value is distinctly less than for the records without preload.

The surge time for the entire spring is measured to be:

T = 10.8 ms.

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- K3 -

5. Discussion of TD-record 70-787 (low, no preload).

Here the impact velocity was reduced to

 $v_0 = 396$ inch/sec.

With $H_{min} = 6.40$ ", the spring does not quite reach solid height. Here the wavehead returns to the front end of spring prior to buffer contact, when the compression velocity is still substantial. After t = 2T, some of the coils clash slightly during the third transition of surge wave, as indicated by the shorter transit time. The wave head is visible up to t = 4T.

During the first transition, the surge time per coil is found to he: $T_1 = c^{248} \text{ ms/coil}$. The entire surge time through the spring is measured to be: T = 11.1 ms.

6. Discussion of TD-record 70-790 (low, with preload).

Here the spring was precompressed to $H_1 = 14.5"$, from a free height $H_f = 16.9"$.

With an impact velocity of 415 inch/sec, the spring reaches a height $H_{min} = 6.40$ " slightly above solid. After t = 2T, the front coils clash, as indicated by the much steeper wave front. The wavehead is visible up to t = 3T-.

During the first transition, the surge time per coil of the wave head is $T_1 = .223$ ms/coil , which is again less than the corresponding value under no preload. The entire surge time was measured to be :

T = 10.5 ms.

L. <u>7-strand M85 drive spring</u>: <u>Discussion of TD-records on sudden release of spring</u>.

1. <u>Review of TD-records taken</u>.

A total of 6 TD-records was taken and delivered as follows:

- Li -

Operating	conditions:	No.of record	No.of record	
Preload	Support of rear end	in report	in second set	
high	rearward only	793*	794*	
medium	rearward only	795*	796*	
medium	fixed both ways	797*	798*	

All TD-records on fast film.

2. Discussion of TD-record 70-793*.

Here the spring was precompressed to an initial height $H_2 = 9$ inches , as compared to a solid height of 6.2". More accurately, the level of precompression is measured by the pitch of the coils, which was found to be $h_2 = .202^{"}$ over 40 coils.

Here the spring had still the original free length $H_f = 18.26"$, or

 $h_{\rho} = .398^{\circ}$ per coil ,

since these sudden release tests were conducted prior to the impact loading tests. This means an initial deflection per coil

f₂ = .398 - .202 = .196" .

The front end of spring (released end) is found to extend at a constant velocity $v_0 = 79\%$ inch/sec. The rear end lifts from its one-way support at a time

 $t = T_s = 11,55 \text{ ms}$

after the sudden release of the front end and then also assumes a constant extension velocity $\mathbf{v}f$ same amount $\mathbf{v}_{\mathbf{a}}$.

The spring then bounces out between two stops. The superposed disturbance originating from the rear end at time $t = T_s$ is due to the wave head flattening out from its original step-form. 3. Discussion of TD-record 70-795*.

Here the spring was released from an approximate height $H_2 = 12$ inches .

- L2 -

More accurately, the level of preload was measured by the pitch of the precompressed coils: $h_2 = .266$ " over 40 coils. With a pitch at free height $h_{f'} = .398$ ", the deflection per coil in the precompressed state then follows to be:

f₂ = .398 - .266 = .132 "/coil.

The extension behavior of the spring is the same as when released from a higher preload.^Here a constant extension velocity

$$v_{a} = 543$$
 inch/sec

was measured.

The separation time T_s of the rear end was found to be:

 $T_{g} = 11.2 \text{ ms}$.

4. Discussion of TD-record 70-797*.

or

Here the first and second end coil (this spring has open ends) were clamped to the guide rod_g so the number of active coils was reduced by at least one coil to about 44 coils.

Here the pitch in the precompressed state was

 $h_{2} = .263$ "/coil over 40 coils ,

i.e., an initial deflection $f_2 = .135"$.

The free end of spring is found to extend with a constant velocity

 $v_0 = 562$ inch/sec.

The spring performs a periodic vibration with the period

4T = 49 ms,T = 11.25 ms.

M. <u>7-strand 20 mm gun drive spring</u>: <u>Discussion of TD-records on impact loading</u>.

1. Review of TD-records taken.

These TD-records on impact loading are numbered from 70-701 to 70-720, with the following records being delivered:

Operating condit:	ions:	No.of record	No.of record	
Impact velocity	precompression	in report	in second set	
nigh	none	709*	710*	
high	yeş	712*	711*	
low	none	714*	713*	
low	yes	717*	716*	
low	yes		7Ì5*	

TDR 70-715* is an odd record, where the precompressing means of spring was released slightly prior to the actual impact of the mass.

Note : All the records delivered were taken at rather low magnification

q = .229x, or 1:4.37 roduction,

in order to accomodate the entire length of spring $(H_f = 29.3")$ within the film width of 8". Therefore, fast film had to be used instead of conventional photo-recording paper.

2. Discussion of TD-record 70-709*(high, no preload).

With a heavier impact mass of 3.25 lbs and an impact velocity

 $\mathbf{v}_{0} = 535$ inch/sec ,

the spring reaches a minimum height

H_{min} = 1.1.6" ,

as compared to a solid height of about 8.5". Therefore, this strong spring is capable of stopping the impact mass of its own and no additional buffer was installed.

When the wave head reaches again the front end of spring at time t = 2T, the compression velocity is already much reduced and the third wave amplitude superposed is much less than that of the wave head. The coils come close to solid, but do not clash, as indicated by the third transit time being of normal length.After t = 3T, however, the rear coils clash, as shown by the much steeper wave front. The wave head is visible up to t = 4T.

During the first transition, the surge time per coil of the wavehead was found to be $T_1 = .455 \text{ ms/coil}$, with the entire surge time T = .17.6 ms.

3. Discussion of TD-record 70-712* (high, with preload).

- M2 -

Here the spring is preloaded to a height $H_1 = 24.1$ ", as compared to its free height $H_p = 29.1$ " (5" deflection).

With an impact velocity $v_0 = 554$ inch/sec, the spring reaches a height $H_{min} = 11.2$ inches . A few rear coils clash already after t = T and the front coils clash after t = 2T, when 3 wave amplitudes superpose to the preload. The wave head is visible up to t = 3T.

The surge time is measured to be: T = 17.5 ms.

4. Discussion of TD-regord 70-714* (lowgno preload).

Here the impact velocity has been reduced to

 $v_0 = 402$ inch/sec.

Here the spring is found to stop the impact mass at a minimum height $H_{min} = 16.3$ inches ,

which is way above its solid height of about 8.5 inches. The coils never clash during their response to impact and the wavehead is visible up to t = 3T.

The surge time is measured to be T = 18, 1 ms.

5. Discussion of TD-record 70-717* (low, with preload).

Here the spring has been precompressed to a height

 $H_1 = 20.4$ inches,

i.e., a considerable initial deflection $F_1 = 8.7$ " from its free height of 29.1".

With an impact velocity $v_0 = 408$ inch/sec, the spring reaches a minimum height $H_{min} = 14.3$ inches. The wavehead reaches the front end of spring at time t = 2T, when the compression velocity is about 0.No clashing of socies occurs and the wavehead is not visible after t = 2T.

The surge time is read to be T = 16.9 ms , i.e., distinctly less than in record 70-714* without preload.

N. Attenuation of single surge wave in stranded wire spring.

N1

The impact of a mass on a spring generates a single surge wave with an infinitely steep wavehead.Both the velocity \mathbf{v} a, the potential u of the end coils jump suddenly from 0 to a certain \mathbf{v}_a and \mathbf{v}_a .

<u>Diagram 4</u> on page Y8 of Appendix E illustrates the felocity distribution v(x,t) over the length L of the spring at various instants of time t = 0, T/4, T/2, 3/4. T, and t = T. The potential distribution u(x,t) is similar. This diagram illustrates in a general manner the change of the wave shape during the first transition through the spring, as shown by all the impact records taken. This <u>first</u> <u>transition of the impact wave</u> through the spring is best suited to study the basic attenuation behavior, since the wave propagation is not disturbed or camouflaged by wave superpositions as yet.

At time t = 0, the wavehead is entering the spring with a vertical step, as indicated by the dotted line $\mathbf{v}(\mathbf{x}, \mathbf{0})$.

At time t = T/4, the peak of the wave has reached the quarter point x = L/4, while the wavehead is already at x = .275.L and the step has changed into a steep slope. The spring coils within this slope are still being accelerated.

At time t = T/2, the wave peak has reached the half point x = L/2. The head of the wave is at x = .55.L, the slope of the wavehead has decreased and the wavehead comprises now more coils still being under acceleration.

As the wave progresses further, the wave head comprising all the coils still under acceleration gets longer and longer; as the distributions at t = .75. T and T show. The beginning of the wavehead has a shorter surge time than the wave peak, e.g., in Diagram 4:

 $T_h = .91 \circ T$

Therefore, we must clearly distinguish between the surge time of the wave peak (T) and the surge time of the wavehead (T_h) . The same applies to the surge times per coil, T_1 and T_{1h} . The surge time per coil T_1 listed on all the records and evaluated therefrom is actually the surge time of the wavehead, but the second subscript was omitted for simplicity reasons.

• 39



The smoothening-out of the wave front during propagation is typical for all stranded wire springs. This behavior has already been observed previously on single wire springs, but to a much lesser degree. It is caused by the damping inherent in the spring and is a welcome feature, since it aids in distributing uneven impact loads more and more evenly over the length of spring, so that all the coils share about equally well in the total deflection of the spring.

Behavior of surge waves after the first transition:

- N2 -

After the first transition, several single surge waves superpose in the spring. However, the wavehead is visible in the TD-records over a length of time from 3T to 4T. Under preload, the wavehead disappears faster, after from 2.5T to 3T.

For sing > wire springs it has been found previously, that the wavehead is discarnible over 6 to 8 surge times T, due to the lesser damping inherent in these springs.

On the other hand, it might be mentioned here, that in highly damped ring springs the wavehead can only be distinguished up to about 1.5.T. Therefore, with respect to wave attenuation, stranded wire springs lie somewhere in the middle between single wire springs and ring springs.

Quantitative measurement of damping:

The above statements about the attenuation of surge waves in stranded wire springs are largely of a qualitative nature. In order to arrive at a quantitative criterion or indicator, standard tests should be defined for determining the degree of damping: E.g., for a suddenly released spring with rear end fixed, the decay of the amplitudes in compression and extension could be measured and a damping constant could be derived therefrom. However, such standard tests and methods are not established as yet, not even foer highly damping spring types as the ring spring. 0. Comparison of measured and calculated surge times T .

One of the major objectives of this study is to obtain <u>realistic formulas for stranded wire springs</u> and to establish the <u>difference in dynamic response</u> as compared to single wire springs. For the latter, a complete set of formulas is available since 20 years and these formulas were well proven by many TD-recordings under a variety of operating conditions.

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Calculation of surge times T :

<u>Table II</u> on page Y2 of Appendix Y lists the surge times T of the 5 test springs as calculated under two different assumptions: Assumption A : The spring rate R is the algebraic sum of the rates of the component single wire springs.

Assumption B : The "initial spring rate" of the measured forcedeflection diagram P(E) [Linear portion] has been used.

According to formula (26), the surge time T increases in propertion to the square root of the spring rate R :

$$T \approx 1/\sqrt{R}$$

Therefore, it must be expected, that the surge time T will decrease with increasing levels of spring compression.

Measured values of surge time T :

<u>Table III</u> on page Y3 of Appendix Y lists the operating conditions for 40 TD-recordings of impact loading of the 5 test springs. These data show the intensity of impact in terms of the initial deflection per coil, f_1 , and impact velocity \mathbf{v}_{α} .

The measured surge times from these 40 impact loadings as well as from 18 sudden release tests are compiled in <u>Table IV</u> on page 74 of Appendix Y.

These measured data have also been plotted in <u>Chart I</u> on page Y5 of Appendix X, for best visual illustration of the results. This Chart is well suited to determine the influence of various parameters, e.g.,

level of impact velocity , v_o , preload versus no preload ,

sudden release tesus versus impact loading.

Influence of the impact velocity level on surge time:

- 02 -

It appears, that in most cases the surge time T is slightly greater with the lower impact velocity than with the higher one. However, this is not absolutely conclusive, since the differences are quite small and also erratic. This uncertainty is quite plausible, since the 2 levels of impact velocity v_0 are not vastly different (see Table III for operating conditions). Recordings at rather different velocity levels, say 100, 300, and 500 inch/sec are believed to shed more light on the influence of v_0 on the surge time T.

Influence of the preload versus no preload:

Chart Î shows clearly, that the surge times T decrease with increasing preload and compression level. These results confirm , formula (22) : With increasing preload, the average spring rate also increases due to the progressive, non-linear force-deflection characteristics, and a shorter surge time T must be expected:

<u>Sudden rélease tosts versus impact loadings:</u>

With the 3 springs tested, the surge time T in sudden release was always found to be a few percent higher than in impact loading. This result is also quite plausible: In extension, the direction of the frictional forces is reversed as compared to compression and a lower force-deflection characteristics P(F), with a lower rate R', results. According to formula (26), a longer surge time T is then to be expected.

Comparison of calculated and measured values of T:

In Chart 1, the T-value calculated under assumption A is indicated by a doted line and the T-value resulting from assumption B by a solid line.

The T-values (A) agree well with the measured values for the two 3-wire strand springs, but are substantially greater than the measured values for the 3 7-wire strand springs.

The T-values (B) <u>agree fairly well</u> with the measured values for all 5 springs. Only with the SPIW-7 and M85-7 springs, the measured values with preload lie still below the calculated values and suggest, that a still higher spring rate than the initial rate should be used for closer agreement.

– P1 –

P. <u>Comparison of measured and calculated extension velocities</u> V_{o.} in sudden release tests.

Again, the objective is to <u>establish a formula</u> to predict the extension velocity v_0 in sudden release from the given precompression, for stranded wire springs in particular.

Measured values of extension velocity:

For 3 test springs,6 TD-records each were taken in sudden release of spring, with 2 records for each operating condition. It was found, that the extension velocities of these 2 records varied very little. In <u>Table V</u> on page Y6 of Appendix Y, the average extension velocity \mathbf{v}_{e} evaluated from these 2 records is listed. The range is

from 416 to 831 inch/sec. Furthermore, the initial deflection per coil, $f_{2^{\circ}}$, of the precompressed springs is also given in Table V:

Calculated values of extension velocities:

Here the formula

 $v_0 = f_2/T_1$

(28a)

was used.T₁ depends on the spring rate R.Again, assumptions A and B are made and the T₁-values obtained under these assumptions are also listed in Table V (top).Finally the $v_0 \leftarrow values$ obtained from above formula (28a) are given in Table V.

Comparison of measured and calculated values.

In Table V, the ratios of measured (to calculated velocity are also listed.Furthermore, for best visual influstration, the v_0 -values have been plotted in <u>Chart II</u> on page 17 of the Appendix Y.

With the <u>3-wire strand</u> SPIW spring, the measured velocity v_0 is about <u>5% less</u> than the value calculated by either method, A or B.

With the two <u>7-wire strand</u> springs, only the method B gives good agreement: Here the measured value v_0 is <u>4.3%</u> and <u>1.3%</u> higher than the value calculated under assumption B. This latter result is quite plausible: Due to the progressive characteristics P(F), the precompressed spring contains actually a higher potential energy than indicated by the initial spring rate R. - Q1 - ~

Q. Summary of major conclusions.



1. Organisation of Final Report.

The text of this report has been supplemented by data and illustrations in 3 Appendices:

<u>Appendix X</u> gives basic spring data in the form of data sheets and measured Force-deflection diagrams (given data).

<u>Appendix Y</u> gives the summary of both calculated and measured spring response data in both tabular and graphical form.

<u>Appendix 2</u> contains 30 samples of time-displacement records (altogether 60 records taken and evaluated) and because of its volume is submitted in a separate cover.

2. Comments on spring rates.

Stranded wire springs are found to have a non-linear, progressive force-defloction characteristics P(F), as illustrated by the measured P(F)-diagrams 1 to 3 in Appendix X.

It appears proper to distinguish between 3 force-deflection rates R = P/F:

- a) The <u>measured initial rate</u>, as defined by the linear initial portion of the diagram, over the first few inches of stroke.
- b) The <u>measured</u> average rate for a certain stroke extending well into the progressive portion of the diagram.
- c) The <u>calculated rate</u>, calculated under the simplifying assumption, that the total rate is the algebraic sum of the spring rates of the single wires in the strand.

From a comparison of these rates it was found, that even the measured initial rate is always higher than the calculated rate, showing an increase of a few percent for a 3-strand spring and increases up to 20% for a 7-strand spring. With large deflections close to solid height, the measured average rate is increased by still larger percentages.

3. Basic behavior of surge waves.

In gener.l, stranded wire springs were found to respond to dynamic loading (impact loading or sudden release) in the same fashion as single wire springs. The surge waves generated travel through the spring at a constant surge velocity and also superpose in the same manner as known from single wire springs. However, there are also marked differences, as the next paragraphs will show. 4. Influence of the deflection or preload on the surge velocity.

- Q2 -

In single wire springs with linear spring rate, the surge velocity c was found to be the same for all levels of spring compression. In the stranded wire springs tested, however, the surge velocity was found to <u>rise slightly</u> with higher loads and deflections. This result is quite plausible from the theoretical viewpoint : The theory predicts that the surge velocity increases in proportion to the square root of the spring rate R.

5. Smoothening of the wavehead.

In a stranded wire spring, the initially infinitely steep slope of a single surge wave generated by impact will <u>decrease</u> in transition and will <u>sprend</u> over <u>several</u> coils by, the time the wavehead reaches the rear end of spring. This response is due to the meachanical and friction coupling between the wires of the strand and is much more pronounced than in single wire springs.

Due to this behavior, the surge time of the wavehead (first response) is shorter than the surge time of the wave peak (where the wave has reached its full amplitude). Therefore, it becomes necessary to distinguish between these two surge times.

6. Disappearance of the wavehead.

In the TD-records on impact loading, the wavehead is visible over a time period of from 2T to 3.5.T after initiation. This time period is much shorter than with single wire springs of comparable length and surge time, where the wavehead is discernible up to 6T or 8T after initiation.

This behavior is caused by the pronounced smoothening of the wavehead during transition due to hogher inherent damping. This response must be considered favorable and desirable, since it aids in making all the coils to share about equally in the total deflection of the spring. The peak stress is thus kept as low as possible, resulting in longer spring life.

7. Formula for surge time and surge velocity.

- 03 -

In the surge time formulas (11) and (12) for single wire springs, the wire diameter d and the mean coil diamèter D enter as the major independent variables.

For stranded wife springs, it has been found more practical to use a less explicit form, because of the more complex configuration. In formula (26), the active weight W of the spring and its rate R enter as independent variables, rather than d and D.Here the question arises, which one of the 3 spring \pm tes listed in Paragraph Q2 above should be used in order to arrive at realistic values for the surge time T .

This question was decided by the comparison of measured surge times (from the TD-records) with calculated surge times (using different values for R .It was found that the <u>measured initial rate</u> gives rather good agreement between measured and calculated surge times T.There is some indication, that sometimes a still higher value of R (toward the measured average rate) would improve the agreement.See <u>Chart I</u> in Appendix Y.

8 . Formula for the extension velocity in sudden release.

According to formula (28) and (28a), the extension velocity v_0 is directly proportional to the precompression as expressed by the initial deflection per coil, f_2 , and inversely proportional to the surge time per coil, T_1 . Again the question arises, by which spring rate R this surge time per coil T_1 should be calculated. And again it is found, that fair agreement between measured and calculated values of v_0 is obtained by using the <u>measured initial spring rate</u> R. A higher spring rate toward the measured average rate would still improve this agreement.

9. Formula for induced load or force.

6

Here formula (29) applies.Judging from the above results, again the intial measured spring rate R should be used for close results. Of course, since no dynamic forces were measured, there was no direct comparison possible.However, when calculating the force from the measured deflection in the records, fair agreement was reached again.



10. Comments on the time-displacement records.

It was found more difficult to obtain good recordings on these 5 stranded wire springs than on single wire springs recorded previously.

On the one hand, the thinner wires, which also lend a much shorter length of wire suitable for light reflection, must be expected to give a less intense and poorer image point than the coil of a single wire spring.

On the other hand, due to the great length of some of the springs, the recordings had to be made at reduced scale, from about .25x to .40x. In certain cases, especially with the 20 mm-7 spring and also in sudden release recording, fast film was used instead of the slower photo-recording paper.

In general, the records obtained show many fine details and the results evaluated from them confirmed the above drawn conclusions again and again. These records contain a wealth of information and may still be evaluated further with other objectives in mind.

Special attention was given to comprehensive and thorough reporting of the operating conditions for each record, to make the evaluated results most meaningful.

Furthermore, the evaluations of times.velocities, etc., were performed as accurately is possible, in order to bring out the sometimes small but still definite differences in dynamic response to that of single wire springs.

R. Recommendations.

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1. Influence of the compression level on surge velocity.

- R1 -

This influence could be checked out over the full compression range by specially devised test recordings: Only 7-strand springs need be used, since they show the most progressive P(F)-diagram. Here the test spring would be precompressed to various ratios $f_1/f_s = 0$, .25, .50, .75 and the spring would be given a limited compression stroke by impact, of less than 2T duration (to avoid more than 2 wave superpositions). The impact velocity would be low, say about 120 inch/sec. These records would then be evaluated regarding surge time/coil of the wavehead and wave peak. The results would reveal the dependency of T and c over the full range of compression level.

2. Quantitative measurement of damping.

Special tests are to be devised and recordings made to establish a quantitative measurement of the damping inherent in a spring. E.g., a very light impact mass (as light as the spring or lighter), would be used and both the impact velocity and rebound velocity would be evaluated in order to determine the meachanical efficiency rate $\eta = v_0^{12}/v_0^2$, (32)

Or the percentage of actual damping to critical damping could be determined from a free vibration.

3. Attenuation of waves in other media.

In order to obtain a full qualitative understanding of the attenuation process in a linear medium and also to arrive at some simple formulas for quantitative damping, the propagation of impact waves should also be recorded for other depuping springs and materials, such as a Poly-Urethane bar, a stack of rubber discs, a bar of hi-damp silicone, etc..

Karl W. Main.

Dr. Karl W. Maier

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APPENDIX X

- Xi -

Basic spring data.	Page
Data sheet on 3-strand SPIW drive spring	X1
Data sheet on 7-strand SPIW drive spring	X 2
Diagram 1 : Measured force-deflection diagram P(E) on SPIW arive springs.3- and 7-strand	X 3
Data sheet on 3-strand M85 drive spring	X4
Data sheet on 7-strand M85 crive spring	X5
Diagram 2 : Measured force-deflection diagram P(F) on M85 drive spring sy3- and 7-strand	X6
Data sheet on 7-strend 20 mm gun drive spring	X7
Diagram 3 : Measured force-deflection diagram P(F) on 20 mm gun drive spring,7-strand	X8

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F

- X1 -

Basic data on SPIV drive spring, 3-strand wire.

AAI, No. 97384, B53099-10096. Drawing No. : music wire Material °034 Wire size, d Strand diameter, d* (reference) .073 Outer diameter of coils, D (measured) .451 Inner diameter of coils, D_i (measured) .317 .384 Mean coil diameter, D Diameter of inner guide rod .300 11.3 Index of coils, C = D/dclosed Type of spring ends Total number of coils, N_t 76.5 Number of active coils, N_{a} 74 Free height of spring, He 17.14 [H, did not change during test program, no setting] Length of 60 middle coils (60.hg) at free height Hg 13.86 Average pitch at free height, h_e °531 Approximate solid height, H_s 5.7 Force deflection rate, P/F , in lbs/inch: Calculated from standard formula 1.38 Measured over the first 5" of stroke (also by drawing) 1.40

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Basic data on SPIW drive spring.coiled from 7-wire strand.

- X2 -

Material music wire, 00470 Number of outer wires 6 Diameter of outer wires, d .026 Diameter of core wire .029 Diameter of wire strand (reference) °081+ Outer diameter, D, , measured .442 Inner diameter, D_i, measured .278 Mean coil diameter, D .360 Diameter of inner guide rod in testing .266 Spring index of outer wires, C = D/d13.85 Type of spring ends closed Total number of coils, N_{t} (after testing) 60.5 Number of active coils, N_{A} 58 Free height H_e, before and after testing 14.26 / 13.80 Length of 50 middle coils, before and after testing 11.90 / 11.65 Average pitch at free height, h_f, .238 / .233 Approximate solid height, Ha 5.3 Force-deflection rate, P/F , in lbs/inch: Calculated from formula 1.85 Measured over the first 5" of stroke 2.10



- X4 -

Basic data on M85 drive spring, coiled from 3-wire strand.

No drawing of spring, all dimensions measu	red.		
Wire diameter,, d			.065
Strand diameter, d* ,(reference)			.1 40 +
Outer diameter of coils, D _o (average)			700
lnner diameter of coils, D _i			•420
Mean coil diameter, D			•560
Diameter of inner guide rod			375
Index of coils, $C \neq D/d$ (husky spring!)			8.62
Type of spring ends			closed
Free height, H_{f} , before and after testing			18.30 / 18.20
Total number of coils (after testing), Nt	ı		44.5
Number of active coils, N _a			42
Length of 40 middle coils	40.h	=	16.84 / 16.76
Average pitch at free height	h _r	H	.421 / .419
Approximate solid height, H _s			6.8
Force-deflection rate, P/F , in lbs/inch:			
Calculated from formula			10.45
Measured over the first 5" of stroke			11,20

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Basic data on M85 drive spring, 7-strand.

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- X5 -

Spring drawing :	Springfield Armory B 7791254	
Material	Spec MIL-W-13604 : FS 1085A	
Number of outer wires		6
Diameter of outer wires	, d	، 038
Diameter of coré wire, «	1.	.041
Diameter of wire strand	, d* (refèrence)	+ 117 ه
Outer diameter of coils	, D _c (measured)after testing)	₀650
Inner diameter of coils	, D _i (measured)	۰ 41 0
Mean coil diameter, D		<i>"</i> 530
Diameter of inner guide	rod	.375
Index of coils, $C \neq D/d$	(outer wires)	13.95
Type of spring ends		plain
Tötal number of coils, l	N_t (after testing)	45.4
Number of active coils,	N _{{L}	45
Free height, H ₁ , before	and after recording tests	18,26 / 16,90 1
[Spring set considerably	y, partly due to impact compressi	ion to H _s]
Length of 40 middle coil	Is at H _f , before and after tests	15.92/15.04
Average pitch at free he	eight, h _f	، 398/ • 376
Approximate solid height	t, H _s	6.2
Force-deflection rate, 1	P/F, in 1bs/inch:	
Calculated from formula	(sum of single wire springs)	3.29
Measured over the first	3" of stroke	4.0



- 55

- X7 -

MM - TR - 7008-2

Basic data on 20 drive spring. 7-strand;

serves the benefit and here

No spring drawing furnished, all data measured.	
Number of outer wires	6
Diameter of outer wires, d	ŏ067
Diameter of core wire (estimated)	2072 [°]
Diameter of wire strand, d* , reference	.206
Outer diameter of coils, D	1.205
Inner diameter of coil's, D	•795
Mean coil diameter, D	1000
Diameter of inner guide rod used in testing	<mark>،</mark> 750
Index of coils, $C = D/d$ (outer springs, rather slender	r) $14_{e.93}$
Type of spring ends	closed, plain
Total number of coils, N _t (after testing)	36.5
Number of active coils, N _a	34
Free height before and after testing, H _f	29.3
Free height durinh testing (slight temporary set)	29.1
Length of 30 middle coils, before and during testing	24.65/24.5
Average pitch at free height, h, before and during	test .822/.817
Approximate solid height, H _s	8.5
Force-deflection rate, P/F, in lbs/inch:	
Calculated from formula (sum of single wire springs)	6°25
Measured over the first 10" of stroke	7.50



APPENDIX Y

Additional Spring Data :

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Data evaluated from the TD-records or calculated from Theory.

- Yi -

Table I	:	Length of wire in coil, L ₁ , for 5 test springs
Table IÍ	:	Numerical calculation of surge times and surge velocity based on two alternate assumptions
Table III	:	Summary of operating conditions in impact loading tests
Table IV	:	Comparison of surge times, measured and malculated
Chart I	:	Surge time T , measured vs calculated values
Table V	:	Sudden release tests: Comparison of calculated and measured extension velocities
Chart II	:	Extension velocity v, measured vs calculated
Diagram 💰	; ?	Attenuation of wavehead in first transition

Spring	SPIN-3	SPIW-7	M85-3	M85-7	.20mm-7
Wire diameter,d	.034	.026	.065	•038	۰ 067
Mean coil dia,D	•384 •	.360	~5 60	•530	1.00
Index, D/d	11.3	13.85	8.62	13 . 95	14.93
Circumf., AD	1.204	1,0130	1 • 7.52	1.662	3.142
Free pitch,h _f	o 2 31	د 238	.421	.398	. 822
$1 + (h_g/\pi D)^2$	1.037	1.044	1.057	1.057	1.068
$\sqrt{1 + ()^2} - \frac{2}{50}$	1.018	1.022	1.028	1.028	1.034
This is relative 1	ength of co	il hciix i	centerline, c	ompared to	T.D.
Absolute length of coil helix CL	1.225	1.155	1.807	1 :71	3 • 25
No. of turns/coil	10	16	7	15.5	17
length of lay,b	٥367 ،	u434	• 973	»6 <u>6</u> 2	1.15
Radius of spiral,a	0195ء	.0275	.0375	.0395	۰069 5
circumf _{e,} 2 T a	.123	.179	۵235ء	•248	•437
length ratio, b*/b	1.054	1.076	1.045	1.068	1.07
Length L, of wire in one coil	1.290	1.243	1.89`	1.823	3.48
length ratio L ₁ / <i>I</i> D	1.070	1.10	1.075	1.097	1 • 1.08
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Table I : Length of wire in coil, L, for 5 test springs.

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based on 2 alternate assumptions, R calculated or measured.									
Spring	SPIN-3	SPIW-7	M85-3	M85 - 7	20mm-7				
Wire diameter, d	¢0،	.026 .029	٥ 65	.038 .041	.067 .071				
Area A = $.785.d^2$	•00091	.:000 53 .00066	0033 2	•00114 •00132	•003 53 •00396				
Area A of strand	.00273	.00384	.01 00	.00816	<u>,</u> 02 51				
Length of coil, L1	1 <u>.</u> 290	1.243	1.89	1.823	3.48				
Weight of foil, W1	.00099	.00135	₀00535	.00421	.0247				
Number of active coils	7'4	58	42	•45	34				
Active weight W = N.W ₁	.073	•078/	° 225	. 1895	• 84				
First assumption: (A)									
Spring rate R <u>calcul</u> .	1.38	1.85	45 ه10	3.29	6.25				
Rate per coil, R ₁	102	107.2	439	148	212,5				
Surge time/coil, T1,ms	₀1585	. 181	.178	•272	<mark>₀55</mark> 0				
Surge time $T = N \cdot T_1, m$	11.73	10.5	7.47	12.23	18.7				
Surge veloc. $c = L_1/T_1$	8,140	6,870	ʻ10 , 620	6,700	6,330				
Force : $\partial P/v = R_1 \cdot T_1$	01615	.0194	.078	٥0402	.117				
Second assumption: (B)									
Initial rate measured	1.40	2.10	11.20	4.00	7.50				
Rate per coil, R ₁	103.6	121.7	470	180	255				
Surge time/coil, T ₁	₀157	•1695	۰172 •	246ء	<mark>₀5</mark> 01				
Surge time T = N.T ₁	11.6	9.83	7.20	11:06	17.05				
Surge veloc. $c = L_1/T_1$	8,210	7,340	11,000	7,400	6,940				
Force: $\Delta P/v = R_1 \cdot T_1$. 0163	.0206	٥805 ٥	•0442	°1275				

Table II : <u>Numerical calculation of surge times and surge velocity</u>, <u>based on 2 alternate assumptions, R calculated or measured</u>.

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SPIV-	3	SPIV-	SPIV-7		M85-3			20 mm-7	7
W = 1	.15 lbs	¥ = 1	.15 lbs	¥ = 2.	15 lbs	₩ = 2.15 lbs		W = 3.25 1bs	
722	475	743.	485	763	480	782	490	709*R	535
	°231		°535		.421		_ë 384		.817
723R	478	744R	480	764*R	473	783 ` R	481	710*	.553
	•231		232 ه		۰ 4 19		J 376		.817
			á, mai	-					
725R	465	745 ·	4.92	7.66	495	7 ⁸⁵	521	711*	564
	.231	υ	.232		.419	• **	. 376		.817
	.055		•046		。Q61		05 0		191 ،
726	465	746R	485	767*R	494	786*R	474	712 * R	554
	°231		.232		•419		376 ،		.817
	.053	<u> </u>	.047		.043		₀ 056 ,		.147
727	.a. 314	747	300	768	390	78 7 K	396	7.13*	380
	.231		232		.419		.376		.817
•			, em aga 145	••					
728R	317	748R	315	769R	382	788	403	714*R.	402
	. 231		.232		•419		.376	•	.817
						*			
730R	322	749 <u>R</u>	335	770	359	790R	415	716 *	400
	.231		°535		•419		.376		.817
	.056		.054		٥074		٥٥51		253 ٥
731	308	750	325	772*R	420	791	409	717*R	408
	°231		° 232		" 419		،376		.817
	.055		.050		. º045		٥٥51		.256

Table III: Summary of operating conditions in impact loading tests.

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SPÍV-3	PIW-3		SPIN-7		N85-3 N85		N85-7		20 mm-7	
Rec.No	•	Rec.No		Rec.No.		Rec.No. Rec.No.		Rec.No.		
T	Ť ₁	T	T ₁	T	т ₁	T	T ₁	T	T ₁	
722		743		763		782R		709*R		
11.8	. 159	9.4	. 158	7.8	.170	10.8	.230	17.6	•455	
723B		744 R		764 * R		783*B		710*		
12.1	•154	9.6	•158	7.95	.170	11.7	.235	17.7	•47	
725R		745		766		785		711*		
12.0	.155	8.7	.141	· 7.47	.159	10 .75	.219	17.1	-	
726		74 5 R		767*r		786*r		712*R		
11.6	.160	9.0	.139	7.9	°161	10.8	.209	17.5		
727		747		768	7	787R		713*		
11.6	.157	9.9	. 170	7.77	163 ،	11.1	•248	17.9		
728R		748R		769 R		788		#14* B		
12.05	• 16 0	9,85	.163	7.8	.160	11.1	.254	18.1,	100-00 CM	
730R		749 R		770		790R		716*		
11.8	° 160	9.4	.143	7.8	J61	10.5	°55°	17.2	-	
731		75 0		772*R		791		717*R		
11.6	•159	8.65	۰134	7.7	.152	10.6	.214	16.9		
vg •11.8	.159	9.3	د 151	7.77	.162	10,9	•229	17.5	.462	
al#11.5	.157	1 9.83	.1695	7,20	J172	11,06	. 246	17.05	.501	
udden r	elease	tests (some spr	ings up	to one	coil sh	orter):			
733	12.03	751R	9.9	*******		793*R	11.55	*** \$40 Min \$40 AM 440 A		
7.34R ⁺	12.2	753	10.2			794*	11.6			
735R	12.4	755R	10.2			795*R	11.2			
7 36	12.1	756	10.5			796*	11.3			
737	12.2	759R	10.2			797*R	11.25			
738R	12.2	760	10,2			798*	11.2			
erages:	12.2		10.2				11.35			
				والمراجعة والمراجعة والمراجعة المراجعة المنا						

TABLE IV : Comparison of surges times T, and T, measured and calculated.

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15 Surge time **K**.S. Chart I Surge 18 Mea sured 76-14 2 0 8 f 6impret lording 2 SPIW-7. SPIW-3 M85. ()

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- 6/5 -MM - TR - 7008 - F T= VI wige time reasured we calculated uctuen. T from Assumption A Ċ. Ð 0 T from Assumption B M85+3 1485 + 7 20 Horn - 7 63

<u>Comparison of calculated and measured extension velocities.</u>											
	v _o . ~	f ₂ /T ₁	, [inch/sec]•	(28°æ)					
Designation/Spring	SPIW-3	5	SPIW-	•7	N85-7	×					
Assumption	A	В	A .	В	A	В					
Surge time/coil, T ₁ ,ms	₀1585	• * 57	.181	• <u>1695</u>	۰272	°246					
High precompression:											
Measured ext.veloc. 0	831		81	9	79	5					
Deflection/coil, f ₂	•13	38 (•1	31	.196						
Calculated ext.vel. v	871	880	° 7 24	773	721	796					
Ratio meas./calcul. v _o	•954	945 ،	13	1.06	1.10	1.0Ŏ					
Medium precompression:		<u>v</u>			,						
Measured ext.veloc: No	416		433		543						
Deflection/coil, f ₂	.071	Ì	。 071		₀132						
Calculated ext.vel. vo	448	452	392	419	485	536					
Ratio meas./calcul. v _o	.929	920	1.105	1. 03ἔ΄΄	1.12	1.07					
Modium precompression: (rear spring end fixed)	ſ	· · · , · · · · ·									
Measured ext.veloc. v _c	437	7	43	94	56	5					
Deflection/coil, f ₂	.071		₀071		₀135						
Calculated ext.vel. v	448	452	39,2	419	496	549					
Ratio meas./calcul. v _o	。 976	. 967	1.105	1.035	î.14	1.03					
Aver.ratio meas./cal.v _o	.953	•944	1.113	t₀043	1.12	1.013					

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Notice difference in ratios between 3-strand and 7-strand springs.



