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ON THE EIGENVALUES OF COUETTE
FLOW IN A FULLY-FILLED
CYLINDRICAL CONTAINER

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Final Report
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ON THE EIGENVALUES OF COUETTE
FLOW IN A FULLY-FILLED
CYLINDRICAL CONTAINER

by

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SUMMARY

For a stationary flow in a cylindrical container of the Couette type in an outer radial zone, and of zero velocity in an inner radial zone, the normal mode equations are derived. For negative wave numbers in the θ -direction, these equations are found to have a singularity.

The eigenvalues are calculated by initial value methods employing the Runge-Kutta-Gill integration procedure. Values of the dependent function and their derivatives at the singularity are calculated by linear extrapolation coupled with continuity requirements.

Tables of eigenvalues for various slenderness ratios of the cylinder and various radial nodes are given for θ -wave numbers of -1 .

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INTRODUCTION

The object of the following investigation is to determine the eigenfrequencies of a viscous fluid, contained in a rotating cylinder, during the initial spin-up period. In order to make the problem tractable, an inviscid fluid is assumed and the fluid is divided into two zones; an outer radial zone in which a Couette type flow prevails and an interior zone which is static. The initial flow distribution, where the interface between the two zones is assumed to decrease from the outer radius to zero in a quasi-static manner, is assumed to be time independent.

FORMULATION OF THE EIGENVALUE PROBLEM

In the following we are interested in the characteristic frequencies of an inviscid liquid contained in a cylindrical container. The liquid is assumed to have an initial stable motion of the Couette type for an outer radial zone whereas the interior is assumed to be at zero velocity. Thus, let the container be of radius a and height $2c$. Then the stationary flow in the outer zone is given by

$$\begin{aligned} u_o &= 0 & a \geq r \geq b \\ v_o &= a\omega[(r/a) - (e^2 a/r)]/(1 - e^2) & 0 \leq [e = b/a] \leq 1 \\ w_o &= 0 \end{aligned} \tag{1}$$

In the inner zone,

$$\begin{aligned} u_i &= 0 \\ v_i &= 0 & b \geq r \geq 0 \\ w_i &= 0 \end{aligned} \tag{2}$$

where b is the radius of the interface between the static interior and the moving exterior; u , v , and w are the radial, longitudinal and axial components of velocity in a cylindrical coordinate system with the

z-axis aligned along the axis of the cylinder. The origin of the coordinate system is located at the bottom of the cylinder so that the z-coordinates of the end faces are given by 0 and 2c. The velocity components in the inner and outer zones are distinguished by the subscripts i and o respectively.

For convenience, we consider the two zones, namely, the exterior zone of Couette flow, and the interior zone of zero velocity, separately. These are then coupled by boundary conditions imposed at the interface, where the radial velocities and pressures corresponding to the two regions are required to be equal. Considering the exterior zone first, the continuity and momentum equations with the appropriate boundary conditions are:

$$\begin{aligned} \frac{\partial u_o}{\partial t} + (Q_o \cdot \nabla) u_o - v_o^2/r &= - \frac{\partial}{\partial r} \left(\frac{P_o}{\rho} \right) \\ \frac{\partial v_o}{\partial t} + (Q_o \cdot \nabla) v_o + u_o v_o/r &= - \frac{1}{r} \frac{\partial}{\partial \theta} \left(\frac{P_o}{\rho} \right) \\ \frac{\partial w_o}{\partial t} + (Q_o \cdot \nabla) w_o &= - \frac{\partial}{\partial z} \left(\frac{P_o}{\rho} \right) \\ Q_o \cdot \nabla &\equiv u_o \frac{\partial}{\partial r} + \frac{v_o}{r} \frac{\partial}{\partial \theta} + w_o \frac{\partial}{\partial z} \\ \frac{u_o}{r} + \frac{\partial u_o}{\partial r} + \frac{1}{r} \frac{\partial v_o}{\partial \theta} + \frac{\partial w_o}{\partial t} &= 0 \end{aligned} \quad (3)$$

The boundary conditions are

$$\begin{aligned} r = a; u_o &= 0 \\ z = 0, 2c; w_o &= 0 \end{aligned}$$

Moreover, at the interface, we have

$$u_{of} = u_{if}; P_{of} = P_{if} \quad (4)$$

where subscript f refers to the interface.

Substituting the following,

$$\begin{aligned} u_o^* &= u_o/a\omega; v_o^* = v_o/a\omega; w_o^* = w_o/a\omega \\ r^* &= r/at; z^* = z/at; t^* = \omega t; \theta^* = \theta \\ p^*/\rho^* &= p/\rho a^2 \omega^2 \end{aligned} \quad (5)$$

and dropping the asterisk, the stationary flow is given by

$$\begin{aligned} u_o &= 0 \\ v_o &= [r - e^2/r]/[1 - e^2] & 1 \geq r \geq e \\ w_o &= 0 \end{aligned}$$

The boundary conditions now become

$$\begin{aligned} r = 1; u_o &= 0 \\ z = 0, 2c/a; w_o &= 0 \end{aligned} \quad (6)$$

At the interface, they are

$$u_{of} = u_{if}; p_{of} = p_{if} \quad (7)$$

Here u_{if} and p_{if} are the appropriate non-dimensional radial velocity and pressure prevailing in the flow field of the inner zone at the interface.

From the non-dimensional form of Equation 3,

$$\begin{aligned} \frac{\partial P_o}{\partial r} &= \frac{v^2}{r} \\ \frac{\partial P_o}{\partial \theta} &= 0 \\ \frac{\partial P_o}{\partial z} &= 0 \end{aligned} \quad (8)$$

where

$$\begin{aligned} P_o &= p_o/\rho \\ v &= [r - e^2/r]/(1 - e^2) \end{aligned} \quad (9)$$

Integrating,

$$P_o = \int_e^r (v^2/y) dy + P_1 \quad (10)$$

where P_1 is the equilibrium pressure at the interface.

We now assume perturbations due to a radial displacement, η , of the interface from its equilibrium position corresponding to $r = e$.

$$\begin{aligned} u_o &= u'_o \\ v_o &= V + v'_o \\ w_o &= w'_o \\ P_o &= \int_e^r \frac{V^2(y)}{y} dy + P'_o + P_1 \end{aligned} \quad (11)$$

Substituting the perturbations into the dimensionless Euler's equations of motion, and neglecting second and higher order quantities,

$$\begin{aligned} \frac{\partial u'_o}{\partial t} + \frac{V}{r} \frac{\partial u'_o}{\partial \theta} - \frac{2V}{r} v'_o &= - \frac{\partial P'_o}{\partial r} \\ \frac{\partial v'_o}{\partial t} + u'_o \frac{dV}{dr} + \frac{V}{r} \frac{\partial v'_o}{\partial \theta} + \frac{u'_o V}{r} &= - \frac{1}{r} \frac{\partial P'_o}{\partial \theta} \\ \frac{\partial w'_o}{\partial t} + \frac{V}{r} \frac{\partial w'_o}{\partial \theta} &= - \frac{\partial P'_o}{\partial z} \\ \frac{u'_o}{r} + \frac{\partial u'_o}{\partial r} + \frac{1}{r} \frac{\partial v'_o}{\partial \theta} + \frac{\partial w'_o}{\partial z} &= 0 \end{aligned} \quad (12)$$

The boundary conditions become

$$\begin{aligned} r = 1; u'_o &= 0 \\ z = 0, 2c/a; w'_o &= 0 \end{aligned} \quad (13)$$

The boundary conditions due to displacement of the interface are obtained in the following manner:

Let the interface be at

$$r = e + \eta \quad (14)$$

Substituting into the last of Equations 11 and neglecting second and higher order terms,

$$P_{of} = P_1 + P'_{of} \quad (15)$$

At the interface, we have from 14

$$u'_0 = \frac{\partial \eta}{\partial t} \quad (16)$$

where second and higher order quantities in the perturbations have been ignored.

We now consider the interior zone. The Euler's equations of motion and the equation of continuity are once again given by Equation 3 in a non-dimensional form. The boundary conditions are

$$z = 0, 2c/a; w_1 = 0 \quad (17)$$

$$r = 0; u_1, v_1 \text{ and } w_1 \text{ are bounded.}$$

At the interface, they are

$$\begin{aligned} u_{1f} &= u_{of} \\ P_{1f} &= P_{of} \end{aligned} \quad (18)$$

We once again assume perturbations due to radial displacement, η , of the interface from its equilibrium position corresponding to $r = e$.

$$\begin{aligned} u_1 &= u'_1 \\ v_1 &= v'_1 \\ w_1 &= w'_1 \\ P_1 &= P'_1 + P_1 \end{aligned} \quad (19)$$

Substituting in Equation 3,

$$\begin{aligned} \frac{\partial u'_1}{\partial t} &= - \frac{\partial P'_1}{\partial r} \\ \frac{\partial v'_1}{\partial t} &= - \frac{1}{r} \frac{\partial P'_1}{\partial \theta} \\ \frac{\partial w'_1}{\partial t} &= - \frac{\partial P'_1}{\partial z} \\ \frac{u'_1}{r} + \frac{\partial u'_1}{\partial r} + \frac{1}{r} \frac{\partial v'_1}{\partial \theta} + \frac{\partial w'_1}{\partial z} &= 0 \end{aligned} \quad (20)$$

The boundary conditions become

$$z = 0, 2c/a, w'_1 = 0 \quad (21)$$

$$r = 0; u'_1, v'_1 \text{ and } w'_1 \text{ are bounded.}$$

At the interface

$$\begin{aligned} u'_1 &= \frac{\partial \eta}{\partial t} \\ P'_{if} &= P_{of} - P_l \end{aligned} \quad (22)$$

NORMAL MODES

In accordance with the usual procedure of treating characteristic value problems, the perturbations are analyzed into normal modes. In view of the boundary conditions, it is natural to suppose that the perturbations are given by quantities which have a (r, θ, z, t) dependence given by

$$\begin{aligned} u'_0 &= U_0(r) \cos [h_0 \pi az/2c] e^{i(K_0 t + m_0 \theta)} \\ v'_0 &= B_0(r) \cos [h_0 \pi az/2c] e^{i(K_0 t + m_0 \theta)} \\ w'_0 &= W_0(r) \sin [h_0 \pi az/2c] e^{i(K_0 t + m_0 \theta)} \\ P'_0 &= R_0(r) \cos [h_0 \pi az/2c] e^{i(K_0 t + m_0 \theta)} \end{aligned} \quad (23)$$

Where K_0 is a constant (which can be complex), m_0 is an integer (which can be positive, zero, or negative) and h_0 is the wave number in the z -direction (which can be 1, 2, 3, etc.). U_0 , B_0 , W_0 and R_0 are functions of r only.

In an analogous manner, for the interior zone, the perturbation quantities may be assumed to be of the form

$$\begin{aligned}
u'_i &= U_i(r) \cos(h_i \pi a z / 2c) e^{i(K_i t + m_i \theta)} \\
v'_i &= B_i(r) \cos(h_i \pi a z / 2c) e^{i(K_i t + m_i \theta)} \\
w'_i &= W_i(r) \sin(h_i \pi a z / 2c) e^{i(K_i t + m_i \theta)} \\
p'_i &= R_i(r) \cos(h_i \pi a z / 2c) e^{i(K_i t + m_i \theta)}
\end{aligned} \tag{24}$$

where K_i , m_i , and h_i are defined as before. U_i , B_i , W_i , and R_i are once again assumed to be functions of the radius r only.

Substituting Equation 23 into Equation 12, we obtain

$$\begin{aligned}
i(K_o + m_o \frac{V}{r})U_o - \frac{2V}{r} B_o &= -\frac{dR_o}{dr} \\
(\frac{dV}{dr} + \frac{V}{r})U_o + i(K_o + m_o \frac{V}{r})B_o &= -i m_o \frac{R_o}{r} \\
(K_o + m_o \frac{V}{r})W_o &= -\frac{h_o \pi a}{2c} i R_o \\
(\frac{dU_o}{dr} + \frac{U_o}{r}) + i m_o \frac{B_o}{r} + \frac{h_o \pi a}{2c} W_o &= 0
\end{aligned} \tag{25}$$

In a similar manner, substituting Equation 24 into 20, we obtain

$$\begin{aligned}
U_i &= \frac{i}{K_i} \frac{dR_i}{dr} \\
B_i &= -\frac{1}{r} \frac{m_i}{K_i} R_i \\
W_i &= -\frac{i}{K_i} \frac{h_i \pi a}{2c} R_i \\
\frac{U_i}{r} + \frac{dU_i}{dr} + \frac{i m_i}{r} B_i + \frac{h_i \pi a}{2c} W_i &= 0
\end{aligned} \tag{26}$$

Since

$$u'_{of} = u'_{if}$$

we have

$$\begin{aligned} U_1(e + \eta) \cos(h_1 \pi a z / 2c) e^{i(K_1 t + m_1 \theta)} \\ = U_0(e + \eta) \cos(h_0 \pi a z / 2c) e^{i(K_0 t + M_0)} \end{aligned} \quad (27)$$

Inasmuch as θ , z and t are independent variables, we require that K_1 , m_1 and h_1 be equal to K_0 , m_0 and h_0 respectively. Thus, the subscripts for K , m and h will be omitted in what follows. Equation 25 can be simplified to the following

$$\begin{aligned} \sigma \left[\frac{dU_0}{dr} + \frac{U_0}{r} \right] - \frac{m}{r^2} U_0 \frac{d}{dr} (\Omega r^2) = \left[\frac{m^2}{r^2} + \frac{\pi^2 h^2 a^2}{4c^2} \right] i R_0 \\ \sigma^2 U_0 - \frac{2\Omega}{r} U_0 \frac{d}{dr} (\Omega r^2) = \left[2\Omega m \frac{R_0}{r} + \sigma \frac{dR_0}{dr} \right] i \end{aligned} \quad (28)$$

where

$$\Omega = V/r \quad (29)$$

and

$$\sigma = K + m\Omega \quad (30)$$

In a similar manner, Equation 26 becomes

$$\begin{aligned} \frac{U_1}{r} + \frac{dU_1}{dr} = \frac{i}{K} \left[\frac{m^2}{r^2} + \frac{\pi^2 h^2 a^2}{4c^2} \right] R_1 \\ U_1 = \frac{i}{K} \frac{dR_1}{dr} \end{aligned} \quad (31)$$

The boundary conditions are

$$U_0(1) = 0 \quad (32)$$

$$U_1(0) \text{ and } R_1(0) \text{ are bounded}$$

The interface conditions are obtained as follows: Since P_{of} is equal to P_{if} , from Equations 15 and 22,

$$P'_{if} = P'_{of} \quad (33)$$

Substituting for P'_{of} and P'_{if} from Equations 23 and 24 respectively,

$$\begin{aligned} R_1(e) \cos(h\pi az/2c) e^{i(Kt + m\theta)} \\ = R_0(e) \cos(h\pi az/2c) e^{i(Kt + m\theta)} \end{aligned} \quad (34)$$

where we consider $\eta \ll e$.

$$\left[K \left\{ \frac{U_1}{r} + \frac{dU_1}{dr} \right\} = \sigma \left\{ \frac{dU_0}{dr} + \frac{U_0}{r} \right\} - \frac{m}{r^2} U_0 \frac{d}{dr} (\Omega r^2) \right]_{r=e} \quad (35)$$

$$U_0(e) = U_1(e) \quad (36)$$

Thus, the characteristic value problem consists of Equations 28 and 31 with boundary conditions 32, 35, and 36. Since we are interested only in those motions which produce a couple on the casing, interest centers around values of $h = (2j + 1)$, where j is an integer.

Axisymmetric Cases*

Setting $m = 0$ reduces Equations 28 and 31 to the following

$$\begin{aligned} \frac{d^2 U_0}{dr^2} + \frac{1}{r} \frac{dU_0}{dr} - \left[\frac{1}{r^2} + N^2 - \frac{N^2}{K^2} \frac{2\Omega}{r} \frac{d}{dr} (\Omega r^2) \right] U_0 = 0 \\ \frac{d^2 U_1}{dr^2} + \frac{1}{r} \frac{dU_1}{dr} - \left[\frac{1}{r^2} + N^2 \right] U_1 = 0 \end{aligned} \quad (37)$$

$$\text{where } N^2 \equiv \pi^2 h^2 a^2 / 4c^2 \quad (38)$$

The boundary conditions become

$$\begin{aligned} U_0(1) &= 0 \\ U_0(e) &= U_1(e); \quad \frac{dU_0(e)}{dr} = \frac{dU_1(e)}{dr} \\ U_1(0) &= 0 \end{aligned} \quad (39)$$

*The axisymmetric case is of academic interest only.

Case 1, The Narrow Gap Approximation for $m = 0$

In specifying Ω , interest is in those velocity distributions which are realizable in a viscous fluid. Equations 9 and 29 determine the case of primary interest as

$$\Omega = (1 - e^2/r^2)/(1 - e^2) \quad (40)$$

However, as $e \rightarrow 1$, an important simplification in the characteristic value problem given by Equations 37, 39 and 40 is possible provided that $(1 - e)$ is small when compared with $(1 + e)/2$. In this case, Equation 40 may be expressed as

$$\Omega = s \quad (41)$$

$$s = (r - e)/(1 - e) \quad (42)$$

The first of Equation 37 may be expressed as

$$\left[\frac{d^2}{dr^2} + \frac{1}{r} \frac{d}{dr} - \frac{1}{r^2} - N^2 \right] U_o = - \frac{N^2}{K^2} \left[2\Omega r \frac{d\Omega}{dr} + 4\Omega^2 \right] U_o \quad (43)$$

Using the transformation defined by Equation 42, consistent with the "narrow gap" approximation of small $(1 - e)$, equation 43 becomes

$$\left[\frac{d^2}{ds^2} - a_1^2 \right] U_o = - \frac{a_1^2}{K^2} \cdot \frac{2e}{1-e} s U_o \quad (44)$$

where

$$a_1^2 \equiv (1 - e)^2 N^2 \quad (45)$$

By means of the following transformation,

$$x = a_1^{2/3} b^{2/3} \left[\frac{1}{b^2} - s \right] \quad (46)$$

where

$$b^2 \equiv 2e/K^2(1 - e) \quad (47)$$

Equation 44 may be expressed as

$$\frac{d^2 U_o}{dx^2} - x U_o = 0 \quad (48)$$

The general solution of Equation 48 may be given in terms of either Bessel Functions of order 1/3 or somewhat more conveniently in terms of Airy's functions AI(x) and BI(x); thus,

$$U_o = A_o AI(x) + B_o BI(x) \quad (49)$$

where A_o and B_o are the integration constants.

In order to determine U_i , the general solution of the second of the differential Equation 37 may be obtained from

$$\frac{d^2 U_i}{dR^2} + \frac{1}{R} \frac{dU_i}{dR} - \left[\frac{1}{R^2} + 1 \right] U_i = 0 \quad (50)$$

$$\text{where } R^2 = N^2 r^2 \quad (51)$$

The general solution of Equation 50 may be written as

$$U_i = A_i I_1(R) + B_i K_1(R) \quad (52)$$

The boundary conditions in terms of the new independent variable x defined by Equation 46 are

$$U_o(x_1) = 0 \quad (53a)$$

$$U_o(x_2) = U_i(Ne) \quad (53b)$$

$$-\frac{dU_o}{dx} = \frac{N(1-e)}{a_1^{2/3} b^{2/3}} \frac{dU_i}{dR} \text{ at } x = x_2; R = Ne \quad (53c)$$

$$U_i(o) = 0 \quad (53d)$$

where

$$\begin{aligned} x_1 &\equiv a_1^{2/3} b^{2/3} \left[\frac{1}{b^2} - s \right] \\ x_2 &\equiv a_1^{2/3} b^{2/3} / b^2 \end{aligned} \quad (54)$$

From boundary conditions 53d, we have

$$A_1 I_1(0) + B_1 K_1(0) = 0$$

Therefore,

$$B_1 = 0 \quad (55)$$

Substituting the general solutions given by Equations 49 and 52 and the condition given by Equation 55 into the boundary conditions 53a, 53b and 53c, we obtain

$$A_0 AI(x_1) + B_0 BI(x_1) = 0 \quad (56)$$

$$A_0 AI(x_2) + B_0 BI(x_2) = A_1 I_1(Ne) \quad (57)$$

$$\begin{aligned} & - \frac{a_1^{2/3} b^{2/3}}{a_1} \left[A_0 AI'(x_2) + B_0 BI'(x_2) \right] \\ & = A_1 \left[\frac{I_0(Ne)}{I_1(Ne)} - \frac{I_1(Ne)}{Ne} \right] \end{aligned} \quad (58)$$

where the primes refer to differentiation with respect to x . A_0 , B_0 and A_1 may be eliminated from Equations 56, 57 and 58 to give the characteristic equation as

$$x_2^{-1/2} \left[\frac{AI'(x_2) BI(x_1) - BI'(x_2) AI(x_1)}{AI(x_1) BI(x_2) - AI(x_2) BI(x_1)} \right] = \frac{I_0(Ne)}{I_1(Ne)} - \frac{1}{Ne} \quad (59)$$

From the definition of x_1 , and x_2 in Equation 54, we obtain

$$x_1 = x_2 - N(1 - e)/x_2^{1/2} \quad (60)$$

For given N and e , the values of x_1 and x_2 which satisfy Equations 59 and 60 simultaneously are the desired values. However, not all pairs of x_1 and x_2 lead to admissible values of K^2 . From Equation 61 we observe that since

$$K^2 = \frac{2e}{1-e} \cdot \frac{x_2}{x_2 - x_1} = \frac{2e}{N(1-e)^2} x_2^{3/2} \quad (61)$$

and since K^2 is a real number, first of all x_2 must be a positive number. Secondly, $(x_2 - x_1)$ must also be positive. Thus, of all the values which satisfy Equations 59 and 60, we select only those pairs of x_1 and x_2 values which satisfy the conditions that $x_2 > 0$ and that $x_2 > x_1$.

Case ii, The Formal Solution for $m = 0$

In case we wish to consider the complete range of e in $1 \geq e \geq 0$, we must consider Equation 37 without making any approximation. We then have

$$\frac{d^2 U_o}{dr^2} + \frac{1}{r} \frac{dU_o}{dr} - \left[\frac{v^2}{r^2} + \alpha^2 \right] U_o = 0 \quad (62)$$

where

$$v^2 \equiv 1 + \frac{4N^2 e^2}{K^2 (1 - e^2)^2} \quad (63)$$

$$\alpha^2 \equiv N^2 - \frac{4N^2}{K^2 (1 - e^2)^2} \quad (64)$$

Equation 62 would need to be considered with the second of Equation 37 and boundary conditions 39. The general solution of Equation 62 is

$$U_o = \ell_v(\alpha r) \quad (65)$$

where ℓ is a general cylinder function of order v . The boundary conditions require ℓ_v to vanish at $r = 1$. The required solution may be expressed as

$$U_o = M [J_{-\nu}(\alpha) J_{\nu}(\alpha r) - J_{\nu}(\alpha) J_{-\nu}(\alpha r)] \quad (66)$$

where M is a constant.

Defined in this manner, U_o clearly vanishes at $r = 1$. We have further at $r = e$.

$$M[J_{-\nu}(\alpha) J_{\nu}(\alpha e) - J_{\nu}(\alpha) J_{-\nu}(\alpha e)] = A_1 I_1(Ne) \quad (67)$$

Moreover,

$$\left. \frac{dU_o}{dr} \right|_{r=e} = A_1 N \left[I_o(Ne) - \frac{I_1(Ne)}{Ne} \right] \quad (68)$$

The integration constants M and A_1 ; may be eliminated from Equations 67 and 68 to yield the characteristic equation. For arbitrarily assigned ν , one may compute K^2 from the characteristic equation and Equations 63 and 64 for given N and e.

THE GENERAL CASE

Consistent with boundedness of the solutions of U_i and R_i at the origin Equation 31 has the following solution

$$R_i = A I_m(Nr) \quad (69)$$

$$U_i = \frac{i}{K} R_i' = A \left[\frac{m}{r} I_m + N I_{m+1} \right] \quad (70)$$

where A is any arbitrary constant. Equation 28 may be rewritten as

$$U_o' = \frac{1}{\sigma} \left[\left(\frac{m^2}{r^2} + N^2 \right) R_o + \frac{2m}{1 - e^2} \cdot \frac{1}{r} U_o \right] - \frac{U_o}{r} \quad (71)$$

$$R_o' = - \frac{2}{\sigma} \left[\frac{2}{1 - e^2} U_o + \frac{m}{r} R_o \right] + \sigma U_o \quad (72)$$

The boundary conditions become

$$U_0(e) = \frac{A}{K} \left[\frac{m}{e} I_m(\text{Ne}) + NI_{m+1}(\text{Ne}) \right] \quad (73)$$

$$R_0(e) = AI_m(\text{Ne}) \quad (74)$$

$$U_0(1) = 0 \quad (75)$$

where R_0 is redefined as $-iR_0$ of equations prior to Equation 69. The eigenvalue problem now is the following: find those values of K for which the Equations 71 and 72, and the boundary conditions 73, 74, and 75 are satisfied.

Since Equations 71 and 72 are homogeneous, any constant times any solution that satisfies the boundary conditions will also be a solution. Thus, in the eigenvalue problem, we can arbitrarily assign any value to the constant A . For convenience, we set it equal to unity.

NUMERICAL SOLUTION

Since Equations 71 and 72 are too cumbersome for closed solutions, we employ numerical integration methods. The numerical procedure for solving the eigenvalue problem is as follows: for an assumed value of K , we can compute the values of $U_0(e)$ and $R_0(e)$ for a given set of values of m , e , and N . Using these as the initial values, we can integrate Equations 71 and 72 by numerical methods to yield $U_0(1)$. If this value is zero, then our choice of K is indeed the right one. If not, we change our assumed value of k by a selected increment and recompute. In practice, since it is uneconomical, computer time-wise, to calculate a precise zero value for $U_0(1)$, we arbitrarily say that if the absolute value of $U_0(1)$ is less than a prescribed positive small number, then our choice of the k value is the right one.

For negative values of m , Equations 71 and 72 have a singularity depending on whether or not the quantity $1 + (1 - e^2) K/m$ is real or

complex. If the quantity is real, the singularity is at a radius given by

$$r_{\text{singular}} = e / \sqrt{1 + (1 - e^2) K/m} \quad (76)$$

At this radius, the derivatives of U_0 and R_0 cannot be computed from Equations 71 and 72. However, from physical considerations, we require that U_0 and R_0 be continuous at this point.

When a singularity exists, we employ the following procedure: We divide the two zones, $e \leq r \leq r_{\text{sing}}$ and $r_{\text{sing}} \leq r \leq 1$ into N_1 and N_2 intervals, where N_1 and N_2 are selected to be such that the increments in radius for an interval is approximately equal in both zones. Starting from $r = e$, we integrate the Equations 71 and 72 till we reach the last interval of $e \leq r \leq r_{\text{sing}}$. We subdivide this into four smaller intervals and integrate the equations to obtain the values of U_0 , R_0 and their derivatives at the three intermediate points shown in Figure 1. Using these values, by linear extrapolation, we estimate the values of U_0 , R_0 and their derivatives at the singularity. Since we cannot use Equations 71 and 72 at the singularity, using the estimated derivatives, we integrate numerically to the one-fourth point of the next interval. From here on, we revert to the conventional integration procedure using Equations 71 and 72.

The numerical integration procedure used in the computer program for calculating the eigenvalues is the familiar Runge-Kutta-Gill method.

DESCRIPTION OF THE COMPUTER PROGRAM

For computing an eigenvalue, the input data to the program is supplied by two cards. The input map for the program, describing the breakdown of the entries in the cards are shown in Table 1. The third I-field on the first card of each set was originally intended to supply the value of n . Since, this was subsequently generalized to be a

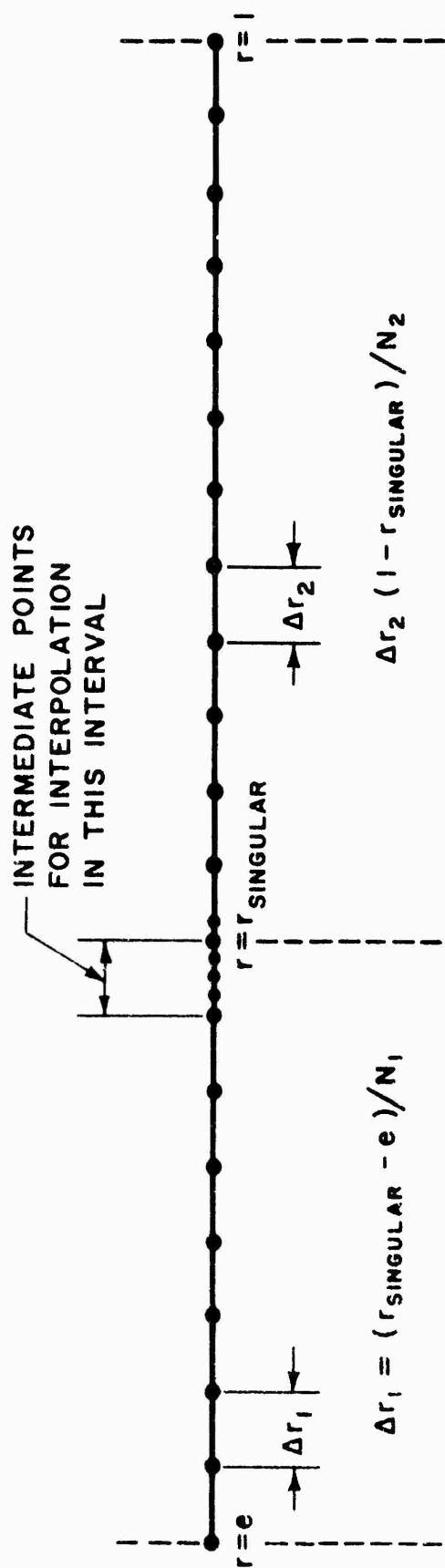


FIG. 1. INTEGRATION INTERVALS

Table 1
INPUT MAP

	<u>Variable</u>	<u>Definition</u>	<u>Format</u>
Card No. 1	M	Run Number	I6
	MM	-m	I6
	NN	Enter zeros	I6
	NR	Number of integration intervals ≤ 500	I6
	IGEN	Number of Eigenvalues required	I6
	NSTOP	If last set of data being read, should be entered non-zero. Otherwise, enter zero.	I6
	LC	$10 N_e \leq 50$	I6
Card No. 2	PLAO	Initial trial value of Eigenvalue	E12.0
	DPLA	Increment of Eigenvalue	E12.0
	TRUNC	Permissible truncation difference	E12.0
	CRAD	e	F12.6

factor in N (CAPN in the program), it may be filled in with a zero or any other number. This number will appear on the printout as the value of n (N in the program). Since it has no other effect on the program, it may be ignored for all other purposes.

The computer program, written in Fortran IV, is given in the Appendix: the first subroutine, NING(NR1) provides the Runge-Kutta-Gill integration method. It also calls a subroutine, DER(Y, FM, CØN, CRAD, FK, AK) which provides the derivatives of U_0 and R_0 as given by Equations 71 and 72. Subroutine NUESTI, called by the main program, provides a method for incrementing the eigenvalue on the basis of current derivation in $U_0(1)$ from the specified truncation error. Subroutine RKGÇØN (AIN, BIN, CIN) supplies the RKG constants to the main program. Subroutine BESCØN(AI, BI), called by the main program supplies the values of the modified Bessel functions. If the program is to be run for any values of m other than -1, this subroutine will need to be changed to supply the modified Bessel functions of the appropriate order.

RESULTS

Table 2 shows the effect of various step sizes on the computed eigenvalues. Truncation is of 10^{-6} , initial eigenvalue is 0.01, eigenvalue increment is .05.

Table 2
EFFECT OF STEP SIZE ON COMPUTED EIGENVALUES

Total No. of Steps	Lowest eigenvalue K for	Lowest eigenvalue K for
	Ne = .2 b/a = e = 0.45 m = -1	Ne = .2 b/a = e = .1 m = -1
100	.76079984	.35125716
200	.75407226	.35046438
300	.75447285	.35032046
400	.75445469	.35025621
500	.75479767	.35021697

The results show that if 200 or more total number of steps are employed, the variation is in the fourth significant digit.

Table 3 lists the results for various Ne and e values computed with 200 integration steps, an initial eigenvalue of 0.01, and an eigenvalue increment of 0.05. With this sweep procedure, the listed eigenvalues were found to be the lowest. It is possible that if the eigenvalue increment is lowered from 0.05 to some smaller value, other, ever lower, eigenvalues may be discovered; but this is considered to be extremely unlikely.

In Stewartson's^{*} notation, we have

$$\frac{c}{a(2j + 1)} = \frac{\pi}{2N}$$

$$b/a = e$$

$$\tau = K$$

* Stewartson, K., *On The Stability of a Spinning Top Containing Liquid*, J. Fluid Mech., 5, 1959, pp. 577-592.

Table 3
LOWEST EIGENVALUES

LOXNe	$\frac{\pi}{2} \frac{a(2j+1)}{c}$	b/a	K = τ	n	LOXNe	$\frac{\pi}{2} \frac{a(2j+1)}{c}$	b/a	K = τ	n
1	2	.05	.35294417	2	3	6	.05	0.10161652	4
	1	.1	.36637878	1		3	.1	0.079672609	2
	.5	.2	.68914656	1		1.5	0.2	0.077654683	1
	.333	.3	.78440604	1		1	0.3	0.38693088	1
	.25	.4	.95493352	1		.75	0.4	0.53024257	1
	.2	.5	.98246679	1		.6	0.5	0.65401489	1
	.166	.6	.99240964	1		.5	0.6	0.93675748	1
	.142857	.7	.99677785	1		.42871	0.7	0.96985291	1
	.125	.8	.99793433	1		.375	0.8	0.98589012	1
	.1111	.9	.99749419	1		.333	0.9	0.99401352	1
2	4	.05	.17450405	3	4	8	.05	0.062517198	5
	2	.1	.35046438	2		4	.1	0.1748773	3
	1	.2	.37567224	1		2	.2	0.33798441	2
	.666	.3	.57999146	1		1.33	.3	0.19826175	1
	.5	.4	.68582012	1		1	0.4	0.39889403	1
	.4	.5	.93226799	1		0.8	0.5	0.5155985	1
	.333	.6	.97111366	1		0.6667	0.6	0.69132881	1
	.285714	.7	.98598000	1		0.571429	0.7	0.94881139	1
	.25	.8	.99387097	1		0.5	0.8	0.97572948	1
	.222	.9	.99698953	1		0.444	0.9	0.98943233	1

n is the number of radial nodes in the range $0 < r < 1$

Table 3 (Concl.)

$10XNe$	$\frac{\pi a(2j+1)}{2c}$	b/a	$k = \tau$	n	$10XNe$	$\frac{\pi a(2j+1)}{2c}$	b/a	$k = \tau$	n
5	10	.05	0.03821307	6	7	14	.05	.094461123	9
	5	.1	0.021487953	3		7	.1	.15244942	5
	2.5	0.2	0.21049394	2		3.5	.2	.24648402	3
	1.666	0.3	0.028242248	1		2.333	.3	.24666304	2
	1.25	0.4	0.27032753	1		1.75	.4	.036690115	1
	1	0.5	0.41369377	1		1.4	.5	.23270392	1
	.833	0.6	0.54600539	1		1.166	.6	.36577586	1
	.714268	0.7	0.9210945	1		1	.7	.53633700	1
	.625	0.8	0.96316761	1		.875	.8	.93412168	1
	.555	0.9	0.98512214	1		.777	.9	.97391060	1
6	12	.05	0.021654819	7	8	16	.05	.076405596	10
	6	.1	0.1034529	4		8	.1	.065089666	5
	3	.2	0.094563088	2		4	.2	.17042725	3
	2	.3	0.32489160	2		2.666	.3	.17456094	2
	1.5	.4	0.14853735	1		2	.4	.32350274	2
	1.2	.5	0.32126868	1		1.6	.5	.14869081	1
	1	.6	0.44177624	1		1.333	.6	.29887522	1
	.857143	.7	0.69269504	1		1.142857	.7	.44032151	1
	.75	.8	0.94963371	1		1	.8	.91714270	1
	.666	.9	0.97912326	1		.888	.9	.96724922	1

n is the number of radial nodes in the range $0 < r < 1$

APPENDIX

A

MAIN PROGRAM FOR THE DETERMINATION OF EIGENVALUES 31G-B2294-01

```

COMMON FM,FN,CON,DEL,NR,X(1,501),Z(1,501),FREQU(3),DET(3),AIN(4),
1BIN(4),CIN(4),CRAD,DPLA,IDET,ILU,KLUE,UCO,RCO,RADIUS(501)
DIMENSION AI(50),BI(50)
NSTOP=0
CALL RKGCON(AIN,BIN,CIN)
CALL BESCON(AI,BI)
20 IF(NSTOP.NE.0)STOP
READ(5,10)M,MM,NN,NR,IGEN,NSTOP,LC,PLAO,DPLA,TRUNC,CRAD
10 FORMAT(7I6/3E12.0,F12.6)
MM=-MM
FM=MM
FN=NN
FNR=NR
FREQU(1)=PLAO
BNC=LC
FNC=BNC/10.
CAPN=FNC/CRAD
CON=CAPN**2
AIM1=AI(LC)*EXP(FNC)
AIM2=BI(LC)*EXP(FNC)
IDET=1
KKK=1
WRITE(6,85)
85 FORMAT(1H1 35HCURRENT VALUES OF INTEGRATION DATA //)
WRITE(6,86)CAPN,LC
86 FORMAT(5X7HCAPN = F10.4,5X5HLC = I2)
DO 30 IG=1,IGEN
ICOUNT=1
70 UCO=(FM*AIM1/CRAD+CAPN*AIM2)/FREQU(1)
RCO=AIM1
RADIUS(1)=CRAD
X(1,1)=UCO
Z(1,1)=RCO
CALL NING(NR1)
DET(1)=X(1,NR+1)
WRITE(6,90)M,IG,FREQU(1),X(1,NR+1)
90 FORMAT(
// 10X 13HRUN NUM
1BER = I3, 5X 5HIG = I2, 5X 16HTRIAL EIGENV. = E15.8, 5X 9H U(1)
2= E15.8)
IF(IDET=2)170,170,180
170 DL = DET(1)
AL = FREQU(1)
GO TO 190
180 IF(AL=FREQU(1))170,170,190
190 ADET = ABS (DET(1))
IF(ADET=TRUNC)80,80,100
100 CALL NUESTI
GO TO (80,110, 20),KLUE
110 IDET = IDET+1
ICOUNT=ICOUNT+1
IF(ICOUNT=50)70,70, 20
80 WRITE(6,1)
1 FORMAT(1H1 41X 32I EIGENVALUES AND EIGENFUNCTIONS )

```

```

WRITE(6,2)M,MM,NN,IG,FREQU(1),CAPN,CRAD
2  FORMAT(//48X 10HR IN NO. = I3// 7X 6H M = I3, 5X 8H N = I3,
1  5X I2, 2X 13HRIGENVALUE = E15.8,9X6HCAPN = F12.6,5X4HC = F6.2)
WRITE(6,3)
3  FORMAT(//18X 2H R 4X 14H U MODE SHAPE 4X 14H R MODE SHAPE 18X 2H
4  R 4X 14H U MODE SIAPE 3X 15H R MODE SHAPE //)
KPA=1 00000047
NRR=NR/2 00000048
DO 40 I=1,NRR 00000049
IF (I=51*KPA) 50,60,50
60 KPA = KPA+1 00000051
WRITE(6,1)
WRITE(6,2)M,MM,NN,IG,FREQU(1),CAPN,CRAD
WRITE(6,3)
50 K=NRR+1 00000055
WRITE(6,130) RADIUS(I),X(1,I),Z(1,I),RADIUS(K),X(1,K),Z(1,K)
130 FORMAT(F20.4, 3X E15.8, 3X E15.8,F20.4, 3X E15.8, 3X E15.8) 00000057
IF(I-NRR) 40,140,40 00000058
140 K = K+1 00000059
WRITE(6,150) RADIUS(K),X(1,K),Z(1,K)
150 FORMAT(40X F20.4,3X E15.8, 3X E15.8)
40 CONTINUE 00000062
FREQU(2)=AL 00000063
DFT(2) =DL 00000064
FREQU(1)=AL+DPLA 00000065
30 IDET=2 00000066
GO TO 20 00000067
END 00000068

```

```

SUBROUTINE NING(NR1)
COMMON FM, FN, CON, DEL, NR, X(1,501), Z(1,501), FREQU(3), DET(3), AIN(4),
1BIN(4), CIN(4), CRAD, DPLA, IDET, ILU, KLUE, UCO, RCO, RADIUS(501)
DIMENSION AK(3), Y(3), Q(3), AM(3), BM(3)
FK=FREQU(1)
Y(1)=CRAD
Y(2)=UCO
Y(3)=RCO
Q(1)=0.
Q(2)=0.
Q(3)=0.
AK(1)=1.
BING=1.+(1.-(CRAD**2))*FK/FM
IF (BING)3,3,1
1 RSING=CRAD/SQRT(BING)
WRITE(6,2) RSING
2 FORMAT(/ /20X 14HR(SINGULAR) = F9.5)
GO TO 4
3 WRITE(6,5)
5 FORMAT(10X38HSINGULARITY OUTSIDE INTEGRATION DOMAIN )
4 BNR=NR
NR1=BNR*(RSING-CRAD)/(1.-CRAD)
NRM=NR-50
IF(NR1.LT.50)NR1=50
IF(NR1.GT.NRM)NR1=NRM
NR2=NR-NR1
FNR1=NR1
FNR2=NR2
DEL=(RSING-CRAD)/FNR1
DEL1=(1.-RSING)/FNR2
DELTA=DEL/4.
NAB=NR1+2
DO 1000 L=1,NAB
IF(L.GE.NR1)DEL=DELTA
DO 100 JJ=1,4
CALL DER(Y,FM,CON,CRAD,FK,AK)
DO 50 I=1,3
AIKN = AIN(JJ)*(AK(I)-BIN(JJ)*Q(I))
Y(I) = Y(I)+DEL*AIKN
50 Q(I) = Q(I)+3.*AIKN -CIN(JJ)*AK(I)
100 CONTINUE
RADIUS(L+1)=Y(1)
X(1,L+1)=Y(2)
1000 Z(1,L+1)=Y(3)
CALL DER(Y,FM,CON,CRAD,FK,AK)
BU=AK(2)
BR=AK(3)
Y(1)=RADIUS(NAB)
CALL DER(Y,FM,CON,CRAD,FK,AK)
AU=AK(2)
AR=AK(3)
AK(2)=2.*BU-AU
AK(3)=2.*BR-AR
Y(1)=RSING
Y(2)=2.*X(1,NAB+1)-X(1,NAB)

```

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```
Y(3)=2.*Z(1,NAB+1)-Z(1,NAB)
RADIUS(NR1+1)=Y(1)
X(1,NR1+1)=Y(2)
Z(1,NR1+1)=Y(3)
DEL=DEL/4,
DO 80 L=1,4
DO 80 JJ=1,4
IF(L,EQ,1)GO TO 81
CALL DER(Y,FM,CON,CRAD,FK,AK)
81 DO 82 I=1,3
   AIKN =AIN(JJ)*(AK(I)-BIN(JJ)*Q(I))
   Y(I) = Y(I)+DEL*AIKN
82 Q(I) = Q(I)+3.*AIKN -CIN(JJ)*AK(I)
80 CONTINUE
RADIUS(NR1+2)=Y(1)
X(1,NR1+2)=Y(2)
Z(1,NR1+2)=Y(3)
DEL=DEL/4
DO 900 L=NAB,NR
IF(L,EQ,NR)DEL=1,-Y(1)
DO 90 JJ=1,4
CALL DER(Y,FM,CON,CRAD,FK,AK)
DO 60 I=1,3
   AIKN =AIN(JJ)*(AK(I)-BIN(JJ)*Q(I))
   Y(I) = Y(I)+DEL*AIKN
60 Q(I) = Q(I)+3.*AIKN -CIN(JJ)*AK(I)
90 CONTINUE
RADIUS(L+1)=Y(1)
X(1,L+1)=Y(2)
900 Z(1,L+1)=Y(3)
RETURN
END
```


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```
SUBROUTINE DER(Y,FM,CON,CRAD,FK,AK)
DIMENSION Y(3),AK(3)
R2=1./(Y(1)*Y(1))
ALPHA=FM*FM*R2+CON
BETA=2./(1.-CRAD*CRAD)
OMEGA=.5*(1.-CRAD*CRAD*R2)*BETA
SIGMA=FK+FM*OMEGA
AK(2)=(ALPHA*Y(3)+FM*BETA*Y(2)/Y(1))/SIGMA-Y(2)/Y(1)
AK(3)=-2.*OMEGA*(BETA*Y(2)+FM*Y(3)/Y(1))/SIGMA+SIGMA*Y(2)
RETURN
END
```

```

SUBROUTINE NUFTI
COMMON FM, FN, CON, DEL, NR, X(1,501), Z(1,501), FREQU(3), DET(3), AIN(4),
IBIN(4), CIN(4), CRAD, DPLA, IDFT, ILU, KLUF, UCO, RCO, RADIUS(501)
DOUBLE PRECISION F321, F213, FD, AF1, F23, F31, F12, D1F1, D2F2, D3F3, AF2,
IF12, AF3, QD, CC, SR1, FF1, FF2
IF(IDFT-2) 10, 20, 30
10 FREQU(2)=FREQU(1)
FREQU(1)=FREQU(2)+DPLA
DET(2)=DET(1)
NCLUF= 3
GO TO 40
20 DDET = DET(1)* DET(2)
NCLUF = 3
IF(DDET) 50, 50, 60
60 ILU = 2
GO TO 70
50 ILU = 1
70 FREQU(3)=FREQU(2)
DET(3) = DET(2)
FREQU(2) = FREQU(1)
DET(2) = DET(1)
95 GO TO (90,100), ILU
90 FREQU(1)=.5*((FREQU(3)+FREQU(2))-((FREQU(3)-FREQU(2))/(DET(3)-
X DET(2)))*(DET(3)+DET(2)))
GO TO 40
100 FREQU(1)=FREQU(2)+DPLA
GO TO 40
110 GO TO(110,120), ILU
110 DDET = DET(1)*DET(2)
IF(DDET)130,130,140
130 NCLUF =1
GO TO 150
140 DDET=DET(1)*DET/3)
NCLUF = 2
IF(DDET)150,150,70
150 IF(FREQU(3))165,165,300
165 GO TO(175,185), NCLUF
175 FREQU(3)=FREQU(1)
DET(3)=DET(1)
GO TO 95
185 FREQU(2)=FREQU(1)
DET(2)=DET(1)
GO TO 95
300 DF12=ABS ((FREQU(1)-FREQU(2))/FREQU(1))
DF13=ABS ((FREQU(1)-FREQU(3))/FREQU(1))
DF23=ABS ((FREQU(2)-FREQU(3))/FREQU(2))
IF(DF12-.1F-04)312,312,313
313 IF(DF13-.1F-04)312,312,314
314 IF(DF23-.1F-04)312,312,311
312 GO TO(165,50), ILU
311 C1 = DET(1)
C2 = DET(2)
C3 = DET(3)
F1 = FREQU(1)
F2 = FREQU(2)

```

F3 = FRFQU(3)	00000172
F321 = (F3-F21)/F1	00000173
F132 = (F1-F3)/F2	00000174
F213 = (F2-F1)/F3	00000175
FD = F321 + F132 + F213	00000176
AF1 = (D1*F321 + D2*F132 + D3*F213)/FD	00000177
F23 = F2/F3	00000178
F31 = F3/F1	00000179
F12 = F1/F2	00000180
D1F1 = D1/F1	00000181
D2F2 = D2/F2	00000182
D3F3 = D3/F3	00000183
AF2 = (D1F1*(F23-1.0/F23)+D2F2*(F31-1.0/F31)+D3F3*(F12-1.0/F12))/FD	00000184
1FD	00000185
AF3 = (D1F1*(1.0/F2-1.0/F3)+D2F2*(1.0/F3-1.0/F1)+D3F3*(1.0/F1-1.0/F2))/FD	00000186
1/F21)/FD	00000187
QD = AF1/AF3	00000188
CC = 0.5*AF2/AF3	00000189
CCD=CC**2-QD	00000190
IF(CCD)312,312,397	00000191
397 SR1 = SORT (CCD)	00000192
FF1 = -CC+SR1	00000193
FF2 = -CC -SR1	00000194
GO TO(160,400),ILU	00000195
400 ILU = 1	00000196
FRFQU(3) = FRFF	00000197
DE(3) = DET(2)	00000198
FREQU(2) = FRFQU(1)	00000199
DET(2) = DET(1)	00000200
C	00000201
80 IF(FF1-FRFQU(3))500,500,600	00000202
500 FRFQU(1)=FF2	00000203
GO TO 40	00000204
600 IF(FF1-FRFQU(2))700,700,500	00000205
700 FRFQU(1)=FF1	00000206
GO TO 40	00000207
160 GO TO(170,180),NCLUF	00000208
170 FRFQU(3)=FRFQU(1)	00000209
DET(3) =DET(1)	00000210
GO TO 80	00000211
180 FREQU(2)=FRFQU(1)	00000212
DET(2) =DET(1)	00000213
GO TO 80	00000214
120 DDET = DET(1)*DET(2)	00000215
IF(DDET)190,190,200	00000216
190 NCLUF = 2	00000217
IF(FRFQU(3))50,50,300	00000218
200 NCLUF = 3	00000219
GO TO 70	00000220
40 GO TO(210,220,230),NCLUF	00000221
210 DFRA = ABS (FREQU(1)-FREQU(3))	00000222
GO TO 240	00000223
220 DFRA = ABS (FREQU(1)-FREQU(2))	00000224
IF(DFRA-.1F-07)250,250,230	00000225
230 KLUF = 1	00000226
GO TO(260,270,280),NCLUF	00000227

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260 FREQU(1) = FREQU(3)
GO TO 280
270 FREQU(1) = FREQU(2)
GO TO 280
230 KIUF = 2
280 CONTINUE
RETURN
END

00000228
00000229
00000230
00000231
00000232
00000233
00000234
00000235

```
SUBROUTINE RKGCON(AIN,BIN,CIN)
DIMENSION AIN(4),BIN(4),CIN(4)
AIN(1)=1./2.
SRT=SQRT(AIN(1))
AIN(2)=1.-SRT
AIN(3)=1.+SRT
AIN(4)=1./6.
BIN(1)=2.
BIN(2)=1.
BIN(3)=1.
BIN(4)=2.
CIN(1)=AIN(1)
CIN(2)=AIN(2)
CIN(3)=AIN(3)
CIN(4)=AIN(1)
RETURN
END
```

```
SUBROUTINE RESCON(AI,RJ)
DIMENSION AI(50),RJ(50)
AI(1) = 0.0452984
AI(2) = 0.0822831
AI(3) = 0.1123775
AI(4) = 0.1367632
AI(5) = 0.1564208
AI(6) = 0.1721644
AI(7) = 0.1846699
AI(8) = 0.1944987
AI(9) = 0.2021165
AI(10) = 0.2079104
AI(11) = 0.2122016
AI(12) = 0.2152568
AI(13) = 0.2172976
AI(14) = 0.2185076
AI(15) = 0.2190694
AI(16) = 0.2190195
AI(17) = 0.2185528
AI(18) = 0.2177263
AI(19) = 0.2166120
AI(20) = 0.2152693
AI(21) = 0.2137478
AI(22) = 0.2120877
AI(23) = 0.2103230
AI(24) = 0.2084811
AI(25) = 0.2065846
AI(26) = 0.2046523
AI(27) = 0.2026990
AI(28) = 0.2007374
AI(29) = 0.1987773
AI(30) = 0.1968267
AI(31) = 0.1948921
AI(32) = 0.1929786
AI(33) = 0.1910902
AI(34) = 0.1892299
AI(35) = 0.1873999
AI(36) = 0.1856022
AI(37) = 0.1838379
AI(38) = 0.1821076
AI(39) = 0.1804119
AI(40) = 0.1787508
AI(41) = 0.1771244
AI(42) = 0.1755325
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ACTUALLY
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ISUBM+1

ACTUALLY

REDDI

GOOF4

- EFN SOURCE STATEMENT - IFN(S) -

02/09/66

ISUBZERO

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END

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		2b. GROUP	
3. REPORT TITLE ON THE EIGENVALUES OF COUETTE FLOW IN A FULLY-FILLED CYLINDRICAL CONTAINER			
4. DESCRIPTIVE NOTES (Type of report and inclusive dates)			
5. AUTHOR(S) (Last name, first name, initial) Reddi, M. M.			
6. REPORT DATE January 1967		7a. TOTAL NO. OF PAGES	7b. NO. OF REFS
8a. CONTRACT OR GRANT NO. DA-30-069-AMC-686(R)		9a. ORIGINATOR'S REPORT NUMBER(S) Technical Report No. F-B2294	
b. PROJECT NO.		9b. OTHER REPORT NO(S) (Any other numbers that may be assigned this report)	
c.			
d.			
10. AVAILABILITY/LIMITATION NOTICES Distribution of this document is unlimited.			
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14. KEY WORDS	LINK A		LINK B		LINK C	
	ROLE	WT	ROLE	WT	ROLE	WT
Eigenvalues, Viscous Fluid Couette Flow Rotating Cylinders						

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