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TECHNICAL REPORT NO. 1-682

# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

by

Albert F. Dorris



Sponsored by

Defense Atomic Support Agency

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Conducted by

U. S. Army Engineer Waterways Experiment Station CORPS OF ENGINEERS

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July 1965

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Vicksburg, Mississippi

ARMY-MRC VICKSBURG, MISS

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#### FOREWORD

This report was prepared in the Nuclear Weapons Effects Division, U. S. Army Engineer Waterways Experiment Station, under the sponsorship of the Defense Atomic Support Agency (DASA) as part of NWER Subtask 13.010, Response of Buried Structures to Ground Shock. The work was accomplished during the period February 1964 through May 1965. During this time, Mr. G. L. Arbuthnot, Jr., was Acting Chief of the Nuclear Weapons Effects Division, and Mr. W. J. Flathau was Acting Chief of the Protective Structures Branch.

This report was prepared by Captain Albert F. Dorris, CE, and is essentially a thesis submitted in partial fulfillment of the requirements for the degree of Doctor of Philosophy in Civil Engineering to the University of Illinois, Urbana, Illinois.

Directors of the Waterways Experiment Station during the period of this study were Colonel Alex G. Sutton, Jr., CE, and Colonel John R. Oswalt, Jr., CE. Mr. J. B. Tiffany was Technical Director.

#### SUMMARY

This was an experimental investigation into the response of small, shallow-buried (in dense, dry sand and stiff clay), aluminum cylinders to static (15-min rise time), rapid (13 msec), and dynamic (0.3 msec) plane-wave loading up to 500 psi. Th. cylinders had identical outside diameters of 3.5 in. and two thicknesses, 0.022 and 0.065 in. Hence, the cylinder stiffnesses,  $EI/R^3$ , were 1.7 and 45 (d/t = 159 and 54), respectively.

In stiff clay, the overpressure required to cause collapse increased very slowly with increasing depth of burial from zero to the deepest burial, three-quarters of the diameter. The hydrostatic buckling equation,  $P_{cr} = 3 EI/R^3$ , was applicable for the cylinders tested.

In the dense id, the overpressure required to cause collapse increased greatly with increasing depth of burial from zero to one-eighth of the diameter. Below this depth it was not possible to collapse even the most flexible cylinders under the available 500-psi pressure. The hoop compression theory was verified. A ductility factor of about 7 was found to be conservative for cylinders buried at depths greater than oneeighth their diameter in the dense sand.

The recorded strains were nonelastic in many cases and it was shown that large yielding does not necessarily define collapse. Stress and moment were found to be nonlinear functions of overpressure, whereas thrust was generally found to be a linear function of overpressure. The differences between static and rapid loading in the elastic response of the cylinder were found to be small. Diameter changes recorded prior to collapse for the static tests were small, less than 5 percent of the diameter.

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THE GRADUATE COLLEGE

May 11, 1965

I HEREBY RECOMMEND THAT THE THESIS PREPARED UNDER MY

SUPERVISION BY\_\_\_\_ALBERT FRANCIS DORRIS

ENTITLED\_\_\_\_\_RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS

TO STATIC AND DYNAMIC LOADING

BE ACCEPTED IN PARTIAL FULFILLMENT OF THE REQUIREMENTS FOR

THE DEGREE OF \_\_\_\_\_ DOCTOR OF PHILOSOPHY IN CIVIL ENGINEERING

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s/ N. M. Newmark

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† Required for doctor's degree but not for master's.

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# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

BY

ALBERT FRANCIS DORRIS B.S., United States Military Academy, 1959 M.S., University of Illinois, 1963

# THESIS

Submitted in partial fulfillment of the requirements for the degree of Doctor of Philosophy in Civil Engineering in the Graduate College of the University of Illinois, 1965

Urbana, Illinois

#### ACKNOWLEDGMENT

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This thesis is based upon experimental studies conducted at the University of Illinois and the U.S. Army Engineer Waterways Experiment Station (WES). The tests conducted at Illinois were sponsored by the Department of Civil Engineering.

These studies were conducted under the general direction of Dr. N. M. Newmark, Professor and Head of the Department of Civil Engineering, and under the direct supervision of Professor G. K. Sinnamon of the Department of Civil Engineering.

Acknowledgment is made to 1st Lt. A. J. Hendron, Jr., and Mr. W. J. Flathau for their comments and encouragement, and to Mr. W. H. Sadler, Jr., who assisted in all phases of the study at WES.

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# TABLE OF CONTENTS

を載め

.....

1

	Page
ACKNOWLEDGMENT	111
NOTATION	ix
LIST OF TABLES	xiii
LIST OF FIGURES	xiv
CHAPTER 1. INTRODUCTION	1
1.1 Background	1
1.2 Problem Under Study	2
1.3 Objective of the Study	3
1.4 Scope of the Investigation	3
CHAPTER 2. DEVELOPMENT OF PRESENT STATE OF KNOWLEDGE	5
2.1 Culvert, Pipe, and Tunnel Contribution	5
2.1.1 Talbot, Cain, Marston	5
2.1.c. Spangler	6
2.1.3 Watkins	7
2.1.4 Schafer, Barnard, White	8
2.1.5 Meyerhof	9
2.1.6 Large Field Structures	9
2.2 Protective Structures Research	10
2.2.1 Dynamic Theory	10
2.2.2 Static Theory	11
2.2.3 Ultimate Strength Laboratory Tests	11
2.2.4 Nondestructive Laboratory Tests	12
2.2.5 Full Scale Tests	13
2.3 Similitude Studies	13

V

Saturday and

# TABLE OF CONTENTS (CONT'D)

		Page
2.4	Bibliographies and Design Manuals	14
CHAPTER 3.	THEORETICAL CONSIDERATIONS	15
3.2	Definition of Failure	15
3.2	Electic Buckling	16
	3.2.1 Soil Medium Approximated by Water	16
	3.2.2 Soil Medium Approximated by Elastic Support	18
	3.2.3 Soil Medium Approximated by an Elastic Medium	21
3.3	Inelastic Action	21
3.4	Characteristic Ring Parameter	22
CHAPTER 4.	EXPERIMENTAL PROCEDURE	26
4.1	Description of Cylinders	26
	4.1.1 Considerations in Selection of Design	26
	4.1.2 Cylinder Material	27
	4.1.3 Cylinder Geometry	28
	4.1.4 End Conditions	28
	4.1.5 Natural Period of Vibration	28
4.2	Description of Soil	30
	4.2.1 Considerations in Selection of Test Soils	30
	4.2.2 Sangamon River and Cook's Bayou No. 1 Sands	31
	4.2.3 Buckshot Clay	31
4.3	Loading Devices	31
	4.3.1 Illinois	31
	4.3.2 WES	32
4.4	Instrumentation	33

vi

Service Contractor

# TABLE OF CONTENTS (CONT'D)

																Page
	4.4.1 G	eneral.	• • • •	•••	••	•	••	•	• •	•	•	•	•	•	•	33
	4.4.2 I	llinois .		••	•••	•	••	•	• •	•	•	•	•	•	•	34
	4.4.3 W	ES	• • • •	• •	•••	•	••	•	•	•	•	•	•	•	•	35
	4.4.4 S	ources of	Error	•••	••	•	••	•		•	•	•	•	•	•	35
CHAPTER 5.	PRESENT	ATION OF 1	EXPERIME	NTAL	RES	JLTS	5.	•	• •		•	•	•	•	•	36
5.1	Method o	f Present	ation .		•••	•	• •	•	••	•	•	•	•	•	•	36
	5.1.1 C	ylinder C	oding .	•••	••	•	••	•		•	. •	•	G	•	•	36
	5.1.2 T	ables of 3	Data .	• •	•••	•	••	•	• •	•	•	ų.	•	•	•	<b>3</b> 5
	5.1.3 D	ate Plots	• • • •	• • •	• •	•	••	•	••	•	٠	•	•	•	•	37
5.2	Computat	ions	• • • •		•••	•		•	• •	•	•	•	•	•	•	37
-	5.2.1 M	oment and	Thrust	Compu	tat:	ion	•	•		٠	•	•	•	•	•	37
	5.2.2 C	omputer P	rogram	• • •	••	•		•	• •	•	•	•	•	•	•	38
	5.2.3 C	omputatio	n of q	• •	• .	•	• •	•	• •	•	•	•	•	•	•	38
5.3	Mode of 3	Failure .	• • • •	• • •	••	•		•	••	•	•	•	•	•	•	39
5.4	Stress,	Moment, a	nd Thrus	st .	•••	٩	••	•	••	•	•	•	•	•	•	39
	5.4.1 A	Group	• • •	• • •	• •	•	••	•	• •	•	. •	•	•	•	•	39
	5.4.2 в	Group .	• • •	• • •	••	•	••	•	••	•	•	÷	•	•	•	40
	5.4.3 C	Group .	• • •	• • •	•••	•			••	•	•	•	•	•	•	40
	5.4.4 E	Group .	• • •	•••	• •	•		•		•	٠	•	•	٠	•	40
	5.4.5 D	Group (C	lay).	• • •	• •	•	••	•	• •	٠	•	•	•	•	•	11
CHAPTER 6.	ANALYSI	S AND INT	ERPRETA	PION C	F T	est	RES	SUL	rs	•	•	•	•	•	•	42
6.1	Overall	Structura	l Respon	nse .	• •	•	••	•	••	•	•	•	•		•	42
	6.1.1 A	Group (S	angamon	Sand)	).	•	. •	•	• •	•	•	•	•	•	•	42
	6.1.2 B	Group (S	angamon	Sand)	).	•		•		•	•	•	•	•	•	49

いいてきないまたを見ていたかいとう

and the second

1048347

TABLE OF CONTENTS (CONT'D)

	Page
6.1.3 C Group (Sangamon Sand)	50
6.1.4 E Group (Cock's Bayou Sand)	52
6.1.5 D Group (Buckshot Clay)	54
6.2 Diameter Change	55
6.3 Arching Ratio	58
6.4 Ultimate Strength	60
CHAPTER 7. SUMMARY, CONCLUSIONS, AND RECOMMENDATIONS	64
7.1 Summary	64
7.2 Conclusions	65
7.2.1 Cylinders in Dense, Dry Sand	65
7.2.2 Cylinders in Stiff Clay	67
7.3 Recommendations for Future Study	68
REFERENCES	70
Tables	79-99
Figures	.00-166
APPENDIX A. PROPERTIES OF ALUMINUM TUBE MATERIAL	167
APPENDIX B. PROPERTIES OF SANGAMON RIVER AND COOK'S BAYOU SANDS	171
APPENDIX C. PROPERTIES OF BUCKSHOT CLAY	181
APPENDIX D. TRANSDUCERS	102
VIIA	198

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### NOTATION

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a Radius of the intrados of the cylinder A Area of the cross section perpendicular to the ring center line AR Arching ratio b Radius of the extrados of the cylinder  $C_u$  Uniformity coefficient,  $D_{60}/D_{10}$ d Outside diameter of cylinder D<sub>r</sub> Relative density,  $\frac{e_{max} - e}{e_{max} - e_{min}}$ <sup>D</sup>10 Soil grain diameter of which 10 percent of the soil weight is finer Soil grain diameter of which 60 percent of the soil weight D60 is finer e Void ratio,  $\frac{V_v}{V_v}$ Maximum void ratio e<sub>max</sub>  $\mathbf{e}_{\min}$ Minimum void ratio Initial void ratio e o Modulus of elasticity of the cylinder, Young's modulus Е E' Modulus of soil reaction, equal to k R, psi E<sub>s</sub> Modulus of elasticity of the soil Acceleration of gravity g  $G_{g}$  Specific gravity of the solids Thickness of the cylinder wall h I Moment of inertia of the cross section of the cylinder wall per unit length, in.  $\frac{1}{y}$ 

k i	Spring	constant,	load	divided	by	deflection
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- ${\bf k}_{\,\rm a}$  Coefficient of elastic soil reaction, psi per strain
- k\_ Coefficient of soil reaction ("subgrade modulus")
- k Modulus of passive resistance of the enveloping earth, psi per inch of deflection, lb/in.3
- k, Radial elastic support
- K Coefficient of earth pressure at rest
  - & Cylinder length
- M Bending moment at the cylinder crown, constrained soil modulus
- M<sub>cs</sub> Constrained secant modulus of soil
- M\_ Bending moment, M
  - n Buckling mode number or order; number of half-waves
- N. Thrust or normal force in the cylinder, lb/in.
- p Pressure, psi
- p Vertical pressure on a horizontal plane through the cylinder crown

Pm,Pt

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- p Critical buckling pressure in lowest mode for a ring subjected to hydrostatic pressure.
  - P Vertical force, 1b
- P Overpressure on surface of soil, psi
- P Overpressure on surface of soil when cylinder collapsed
  - q Ratio of average horizontal force (or pressure) to average vertical force (or pressure) applied to the cylinder
  - q. Unconfined compressive strength
    - Q Vertical shear force in soil between surface and cylinder crown

- Q' Vertical shear force in soil between cylinder crown and spring line
- Q" Oblique shear force in soil between cylinder crown and spring line
  - r Radius of a cylinder element
  - R Radius of the cylinder middle surface
- S\_ Degree of saturation
- S,S, Relative stiffness
  - t Time
  - $\mathbf{T}_{\mathbf{x}}$  Period of vibration in the compressive mode
  - T, Period of vibration in the first flexural mode
  - TD Typical descriptor of relative stiffness
  - V Total volume of soil sample
  - V Initial volume
  - V Volume of soil solids
  - V Volume of voids
  - w Radial displacement of the cylinder; water content

x,y,z Cylinder coordinates, spatial cordinates

- Z Vertical distance from soil surface to cylinder crown
- 7 Unit weight of soil, specific weight
- $7_d$  Dry unit weight
- A Horizontal deflection (increase in diameter)
- A. Vertical deflection (decrease in diameter)
- **∆V** Volume change
- € Unit strain
- Strain on extrados of the cylinder
- e, Strain on intrados of the cylinder

- $\theta$  Circular angle
- v Poisson's ratio of the cylinder
- $v_s$  Poisson's ratio of the soil
- σ Stress
- $\sigma_{\mathbf{y}}$  Stress in the y or tangential direction
- $\sigma_{yl}$  Lower or first yield stress
- $\sigma_{y2}$  Upper yield stress (result in 0.2 percent permanent strain)
- $\sigma_1$  Vertical stress
- $\sigma_3$  All-around confining stress
- Ø Angle of internal friction

xiii

# LIST OF TABLES

Table Number	Title	Page
4.1	Geometric and Material Properties of Test Cylinders	79
5.1	Overall Testing Program and Overpressure, P <sub>so</sub> , at Failure	80
5 <b>.2</b>	Strain, Stress, Thrust, Moment, and Deflection; Tests A-1, A-2, A-3A, A-3B, A-4, A-5	81
5•3	Strain, Stress, Thrust, and Momment; Tests A-6, A-7, A-8, A-9, A-10	83
5.4	Strain, Stress, Thrust, Moment, and Deflection; Tests B-1A, B-1B, B-2, B-3, B-4, B-5	<b>8</b> 5
5.5	Strain, Stress, Thrust, and Momment; Tests B-6, B-7, B-8, B-9, B-10	87
5.6	Strain, Stress, Thrust, Moment, and Deflection; Tests C-1, C-2, C-3, C-4, C-5	89
5.7	Strain, Stress, Thrust, and Moment; Tests C-6, C-7, C-8, C-9, C-10	91
5.8	Strain, Stress, Thrust, Moment, and Deflection; Tests E-1, E-2, E-3	93
5.9	Strain, Stress, Thrust, and Moment; Tests E-4, E-5, E-6	95
5.10	Strain, Stress, Thrust, Moment, and Deflection; Tests D-1, D-2, D-3, D-4, D-5	96
5.11	Strain, Stress, Thrust, and Moment; Tests D-6, D-7, D-8, D-9, D-10	<b>99</b>
C.1	Pretest Properties of Clay Specimens	186

# LIST OF FIGURES

ł

Figure Number	Title	Page
1.1	Concepts of Load Transfer	100
2.1	Concepts of Load Distribution	101
3.1	Cylindrical Shell and Ring Notation	102
3.2	Actual Modes of Failure	103
3.3	Buckling Modes	103
3.4	Nonuniform Load	103
3.5	Elastic Supports	103
3.6	Elastic Medium	10_
3.7	Idealized Loading Configurations	104
4.1	Longitudinal Section of Cylinder and Gage Locations	105
4.2	University of Illinois Blast Load Generator	106
4.3	Overpressure-Time Relation for Rapid and Dynamic Loading	107
4.4	WES Small Blast Load Generator (SBLG) Facility	108
4.5	Illinois Instrumentation Equipment	108
4.6	WES Large Instrumentation Room	109
4.7	WES Small Blast Load Generator (SBLG) Instrumentation	109
5.1	Stress, Thrust, Moment, and Deflection, Test A-1 $(Z = 0 \text{ in.})$	110
5.2	Stress, Thrust, Moment, and Deflection, Test A-5 $(Z = 3/16 \text{ in.})$	111
5.3	Stress, Thrust, Moment, and Deflection, Test A-2 $(Z = 7/16 \text{ in.})$	112
5.4	Stress, Thrust, Moment, and Deflection, Test A-3A $(2 = 7/8, in.)$	113
5.5	Stress, Thrust, Moment, and Deflection, Test A-3B $(7 - 7/8)$ in.)	114

XV

Figure Number	Title	Page
5.6	Stress, Thrust, Moment, and Deflection, Test A-4 $(Z = 1-3/4 \text{ in.})$	115
5.7	Stress, Thrust, and Moment, Test A-10 ( $Z = 0$ in.)	116
5.8	Stress, Thrust, and Moment, Test A-9 ( $Z = 3/16$ in.).	117
5.9	Stress, Thrust, and Moment, Test A-8 ( $Z = 7/16$ in.)	118
5.10	Stress, Thrust, and Moment, Test A-7 ( $Z = 7/8$ in.)	119
5.11	Stress, Thrust, and Moment, Test A-6 ( $Z = 1-3/4$ in.)	120
5.12	Stress, Thrust, Moment, and Deflection, Test B-1A (Z = O in.)	121
5.13	Stress, Thrust, Noment, and Deflection, Test B-1B $(2 = 0 \text{ in.})$	122
5.14	Stress, Thrust, Moment, and Deflection, Test B-5 $(2 = 7/16 \text{ in.})$	123
5.15	Stress, Thrust, Moment, and Deflection, Test B-2 $(2 = 7/8 \text{ in.})$	124
5.16	Stress, Thrust, Moment, and Deflection, Test B-3 $(2 = 1-3/4 \text{ in.})$	125
5.17	Stress, Thrust, Moment, and Deflection, Test B-4 $(2 = 2-5/8 \text{ in.})$	126
5.18	Stress, Thrust, and Moment, Test B-6 (2 = 0 in.)	127
5.19	Stress, Thrust, and Moment, Test B-7 (Z = 7/16 in.)	128
5.20	Stress, Thrust, and Moment, Test B-8 (Z = 7/8 in.)	129
5.21	Stress, Thrust, and Moment, Test B-9 (Z = 1-3/4 in.)	130
5.22	Stress, Thrust, and Moment, Test B-10 (2 = 2-5/8 in.)	131
5.23	Stress, Thrust, Moment, and Deflection, Test C-1 $(2 = 0 \text{ in.})$	132
5.24	Stress, Thrust, Moment, and Deflection, Test C-4 $(2 = 3/16 \text{ in.})$	133
5.25	Stress, Thrust, Moment, and Deflection, Test C-5 $(7 \pm 5/16 \text{ in.})$	134

xvi

#### Figure Title Number Page 5.26 Stress, Thrust, Moment, and Deflection, Test C-2 135 (Z = 7/16 in.)Stress, Thrust, Moment, and Deflection, Test C-3 136 5.27 (2 = 7/8 in.)5.28 Stress, Thrust, and Moment, Test C-6 (Z = 0 in.)137 Stress, Thrust, and Moment, Test C-7 (Z = 3/16 in.)5.29 138 Stress, Thrust, and Moment, Test C-8 (Z = 5/16 in.)5.30 139 Stress, Thrust, and Moment, Test C-9 (Z = 7/16 in.)140 5.31 5.32 Stress, Thrust, and Moment, Test C-10 (2 = 7/8 in.)141 142 5.33 Thrust, Moment, and Deflection, Test E-3 (Z = 0 in.)Thrust, Moment, and Deflection, Test E-2 (2 = 7/16 in.)5.34 143 Thrust, Moment, and Deflection, Test E-1 (Z = 7/8 in.) 144 5.35 5.36 Strain, Tests E-5 (2 = 7/16 in.) and E-4 (2 = 7/8 in.); 145 Surface Overpressure = 250 psi 146 Thrust and Moment, Tests E-6 (2 = 0 in.), E-5 5.37 (Z = 7/16 in.), and E-4 (Z = 7/8 in.); Surface Overpressure = 250 psi 5.38 Stress, Thrust, Moment, and Deflection, Test D-1 147 (Z = 0 in.)148 5.39 Stress, Thrust, Moment, and Deflection, Test D-2 (2 = 7/16 in.)5.40 Stress, Thrust, Moment, and Deflection, Test D-3 149 $\sqrt{2} = 7/8$ in.) 5.41 Stress, Thrust, Moment, and Deflection, Test D-4 150 (2 = 1 - 3/4 in.)5.42 Stress, Thrust, Moment, and Deflection, Test D-5 151 (2 = 2-5/8 in.)Strain, Thrust, and Moment, Tests D-6 Through D-10 5.43 152 Cylinders of Groups A, B, and C after Tests 5.44 153

からいたいで

122

Figure Number	<u> </u>	Page
5.45	Cylinders of Groups D and E after Tests	154
5.46	Cylinders D-6 and D-10 after Test	154
5.47	Posttest Cylinder Configuration in Clay	155
5.48	Relation Between Failure Pressure and Depth of Burial	156
6.1	A Group: Average Spring-Line Thrust, Crown Moments, Vertical Diameter Changes, and q Values	157
6.2	B Group: Average Spring-Line Thrust, Crown Moments, Verticel Diameter Changes, and q Values	158
6.3	C Group: Average Spring-Line Thrusts, Crown Moments, Vertical Diameter Changes, and q Values	159
6.4	Peak Diameter Changes and Deflection Stiffnesses	160
6.5	Static Arching Ratio	161
6.6	Relation Between Failure Pressure and Cylinder Stiffness	162
6.7	Relation Between Nondimensional Pressure and Equation 3.1	163
6.8	Relation Between Nondimensional Pressure and Equation 3.16	164
6.9	Relation Between Nondimensional Pressure and Equation 3.8	165
6.10	Relation Between Nondimensional Fressure and Equation 3.17	166
A.1	Aluminum Stress-Strein Properties	170
B.1	Sand Flacement in the SBLG	177
B.2	Gradation Curves for the Sands	177
B.3	Stress-Strain Relations for Sangamon River Sand	178
B.4	Stress-Strain Relations for Cook's Bayou No. 1 Sand	179
B.5	Moduli and Strength Characteristics for Sands	180
C.1	Placement of Buckshot Clay in WES Small Blast Lond Generator	187

*****	tvx	11
-------	-----	----

f

Figure Number	<u> </u>	Page
C.2	Placement of Cylinder in Buckshot Clay and Subsequent Backfilling	188
C.3	Average Placement Strains	189
c.4	Gradation Curve for Buckshot Clay	189
C.5	Atterberg Limits	190
c.6	Unconfined Compressive Strength-Water Content Relation	190
C.7	Stress-Strain Relations for Buckshot Clay	191
C.8	Relation Between Moduli and Pressure for Buckshot Clay	192
C.9	Density-Moisture Content Relation for Buckshot Clay	192
D.1	Strain Gage Test	197
D.2	Diameter Change Gage	197

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# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

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CHAPTER 1. INTRODUCTION

### 1.1 Background

The art of designing buried structures to resist nuclear blast loading is still (1965) in its infancy. A desirable way of augmenting the development and evaluation of particular protective structures designs is to conduct full-scale tests; however, the moratorium on full-scale surface tests in effect since 1 November 1958 eliminates this approach in studying the response of shallow-buried structures to overpressure-induced disturbances. Unfortunately, even if full-scale tests had been permitted since 1958, it is doubtful that sufficient data would be available from such tests alone to formulate economical and practical designs for most design situations. Laboratory and analytical studies still would have been needed to supplement such programs. Because of the limitations imposed by the muratorium, special emphasis has by necessity been placed on analytical studies and laboratory tests of small-scale structures for the purpose of developing usable design methods.

At the moment there is a lack of well-documented experimental data and field experience with which to compare current thought and analytical theory. The most advanced design manual, <u>Principles and Prac-</u> <u>tices for Design of Hardened Structures</u> by Newmark and Haltiwanger (1962, under revision),\* and the current source book of underground phenomena and effects of nuclear weapons, <u>Nuclear Geoplosics</u> by Stanford Research

\* Authors and dates refer to list of references at end of text.

Institute (1964), point out a multitude of unknowns in the state of the art. 1.2 Problem Under Study

Buried cylindrical or ring configurations are ideal geometries to resist external loads effectively and are thus well suited to protect personnel and appurtenances for various facilities such as NIKE and ICEM hardened sites. They are also favored as entrances and escape routes for protective structures buried deep in rock. Additionally, almost all communication and utility conduits, existing and planned, are cylindrical in shape. Currently, these structures are being designed largely on the basis of engineering hypotheses supplemented by the field experience gained with buried conduits and tunnel liners subjected to static loading. There is virtually no experimental validation of the current dynamic design criteria. Because of the uncertainties, the current design procedures are only stopgap measures which await the results of controlled experimental investigations for confirmation or refutation.

The problems of designing shallow-buried protective structures for overpressure-induced loading from large-yield weapons differ from those associated with other underground cylindrical structures in at least two major ways: (1) The live load is large compared with the dead load, and the structure must be designed primarily for the live load; (2) the criteria for failure, together with the factor of safety, must lead to the least expensive structure which couples cost and use to fulfill requirements. A factor of safety of 4 is common in culvert design as indicated by Armco Drainage and Metal Products, Inc. (1958, p 70). This factor is sufficiently large to take care of many unknowns. However, a factor this large is economically infeasible for the design of most protective structures.

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### 1.3 Objective of the Study

The objective of this investigation was to study experimentally the phenomena associated with the failure of horizontally oriented, circular cylinders buried at various shallow depths in several soil media and subjected to either static or dynamic overpressures.

# 1.4 Scope of the Investigation

It would be desirable to study a wide range of cylinder types by varying such parameters as material properties of the cylinder, cylinder dimensions, soil media, depths of burial, overpressure characteristics, and combinations of instrumentation transducers. Experimentally, very little ultimate strength work has been done to study buried cylindrical structures in the collapse range.

An evaluation of all the parameters and combinations in detail would be far beyond the scope of any single investigation. The parameters selected for study are outlined below:

- In order to examine the extreme range in soil media, two soils were selected: a dense, dry sand and a highly plastic clay placed at such a water content that the consistency ranged from stiff to very stiff as defined by Terzaghi and Peck (1948, p 31).
- In order to examine the effect of depth of burial,
   five shallow depths, ranging from zero to 2-5/8 in. or
   3/4 diameter (d), were investigated.
- 3. In order to examine overpressure effects, three pressure-time signatures were used, ranging from a quasi-static rise time of 10 to 15 min, to a rapid

rise time of 13 msec, and up to a dynamic rise time of 0.3 msec.

4. In order to examine a range in structural stiffness, two circular cylinder geometries (two wall thicknesses and three nominal yield strengths) were employed. The outside diameter, length, and end conditions were kept constant.

Since underground cy indrical structures have long been used as tunnels, culverts, sewers, and pipes, a great deal of qualitative knowledge is available covering all aspects of the soil-structure system, e.g. arching, longitudinal beam action, live load distribution, ring loading, and ring response. Fig. 1.1 illustrates some of the concepts of load transfer from the soil surface to the underground structure.

This test program was planned to investigate ring response, and the emphasis was not on the associated phenomena such as arching. These will be discussed only as they contribute to an understanding of the ring response.

Forty-six cylinders were tested during the investigation. For each rapid or dynamic test (plane wave loading), a corresponding static test was performed for comparison. The entire program is summarized in Table 5.1.

The 30 cylinders designated as groups A, B, and C were tested under static and rapid loading in the blast-loading facility at the University of Illinois. The 16 cylinders in groups D and E were tested under static and dynamic loading at the U. S. Army Waterways Experiment Station (WES).

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### CHAPTER 2. DEVELOPMENT OF PRESENT STATE OF KNOWLEDGE

### 2.1 Culvert, Pipe, and Tunnel Contribution

It is not the writer's purpose to cite all of the potentially applicable work, but rather to categorize the development of current schools of thought and to summarize the more pertinent references describing the development of design and analysis procedures for buried cylinders.

2.1.1 Talbot, Cain, Marston

Talbot (1908) tested cast-iron, plain concrete, and reinforced concrete pipes to failure. He recognized both the beneficial effect of lateral confinement (p 22) and the ability of the concrete rings to retain their circular configuration until final failure occurred when the concrete crushed (p 65). The idealized load distribution which he considered is shown in Fig. 2.1a. In view of the fact that the load distribution was not uniform, that the actual value of q (the average horizontal pressure divided by the average vertical pressure) was not determinable, and that cracking would not be acceptable for permanent installations, Talbot recommended the use of the formula  $M_c = 0.25p_aR^2$  for design, i.e. the maximum bending moment (at the crown),  $M_c$ , with q = 0 where  $p_a$  is the average pressure on a horizontal plane through the crown, and R the mean radius of the pipe. Any surplus strength offered by the side restraint would be "considered merely an additional margin of safety" (Talbot (1908)).

Braune, Cain, and Janda (1929) explored the possibility that the horizontal pressure was not distributed all the way to the top of the ring (Fig. 2.1b). Using the results of pressure cell measurements on the surface of relatively flexible rings, they (in Appendix II written by Cain)

tried to arrive at applicable values of  $\theta$ , the circular angle, and q. Cain also discussed (p 173) the reasons why deflections determined by a uniform radial load theory would never agree with measured values. This theory treats the horizontal passive soil resistances as if they were active soil forces.

Marston (1930) summarized his own work on arching and gave some guidelines to define the differences between flexible and rigid conduits. He considered flexible conduits as having cross-sectional shapes that can be distorted sufficiently to change their vertical or horizontal dimensions more than 3 percent before causing materially injurious cracks; rigid conduits cannot sustain such distortions.

2.1.2 Spangler

Spangler (1938) used a friction tape technique to measure the pressure distribution on the outside of flexible metal pipes. He developed a hypothetical distribution of pressure, Fig. 2.1c, based on the maximum unit horizontal pressure being equal to the modulus of passive resistance,  $k_g$ , of the fill material multiplied by one-half the horizontal diameter change. Spangler used e for this, but for distinction within this report the term  $k_g$  shall be used. He stated that deflection of a flexible culvert is the phenomenon of primary interest "because failure of flexible pipes occurs by excessive deflection rather than excessive stress." Spangler's design formula (Iowa Formula) for good bedding, Fig. 2.1c, also shows the relative influence of the pipe parameter, EI, and the influence of the passive soil resistance parameter, 0.061  $k_g R^h$ , where E is the modulus of elasticity of the pipe, I is the moment of imertia of the pipe wall, and R is the mean radius of the cylinder.

Spangler (1948) reviewed the state of knowledge of underground conduits and pointed out the lack of knowledge concerning the modulus of passive resistance,  $k_g$ . He also indicated that the load distribution on a horizontal plane at the level of the cylinder crown,  $p_a$ , is approximately uniform over the breadth of the pipe. Spangler (1956, pp 1054-9) discussed the validity of assuming a condition of plane strain or plane stress for pipeline problems. He concluded that it is not possible to determine which most nearly applies, and used the somewhat simpler plane stress assumption which is not dependent upon Poisson's ratio, v, of the cylinder. Spangler (1960, Chapter 25) further discussed the Iowa Formula and tentatively recommended that for flexible culverts the deflection should not exceed 5 percent of the diameter. Typical values for the modulus of passive resistance were mentioned. Spangler indicated that the modulus of passive resistance is strongly influenced by the size of the pipe and gave recommended values for design.

# 2.1.3 Watkins

Watkins and Spangler (1958) examined the Iowa Formula from a dimensional analysis or similitude point of view. It was concluded that the modulus of passive resistance is not a property of the soil alone; and, further, that the product of the modulus of passive resistance,  $k_g$ , times the pipe radius is a constant for a given soil. This quantity,  $k_g R$ , was termed the modulus of soil reaction, E'.

Watkins (1959) attempted to correlate the modulus of soil reaction to properties that are easily measured. His work indicated that the modulus was related to the compression index for a given soil. Watkins (1960) pointed to buckling of the pipe wall, before an excessive diameter

change has occurred, as a potential failure mechanism for buried conduit systems. Watkins (1963) suggested that the hydrostatic buckling equation,  $p_o = \frac{3EI}{R^3}$  (where  $p_o$  is the critical buckling pressure in psi), be applied as a conservative estimate of the buckling failure phenomenon. This and the work of Brockenburgh (1963) influenced the U. S. Steel Corporation to produce a new corrugation profile for their flexible culverts. Watkins and Nielson (1964) developed a test apparatus, modpares device, to measure the modulus of soil reaction. It was found that this quantity is not a constant, but rather decreases with increasing conduit deflection.

Watkins (1964) again pointed out the importance of the soil in influencing structural response, and illustrated the possibility of buckling for a very flexible ring carefully embedded in a well-compacted, granular fill.

2.1.4 Schafer, Barnard, White

Schafer (1948) stated that an average safe maximum deflection for conduits is 20 percent of the vertical diameter. Application of a factor of safety of 4 to the deflection criterion leads to a design deflection of 5 percent. He developed an empirical deflection equation, examined the lowa Formula, and concluded that it gave undue value to the side-support factor,  $k_g$ , for large-diameter structures.

Barnard (1957) pointed out that apparent bending stresses in steel pipe based on elastic theory are not of importance in themselves when the ductility of the material in the shell permits deformation without failure. Localized bending stresses which appear to pass the yield point of the material are not proper criteria for failure.

White and Layer (1960) proposed the ring compression theory,

Fig. 2.1d, as a rational design tool. They argued that the ring bending stiffness need only be sufficient (1) to prevent buckling, (2) to resist the uneven loads in minimum cover installations, and (3) to permit easy handling and erection. White (1961) described a 21-ft-diameter corrugated culvert designed by using the ring compression theory, and indicated that the primary factor for average corrugated metal conduits is compressive strength.

# 2.1.5 Meyerhof

Meyerhof and Baikie (1963) performed tests to failure on quarter sections of curved steel sheets bearing against dense sand backfill. They showed that for small values of the subgrade modulus and the flexural rigidity of the plates, the sheets would fail by buckling; but, for larger values of these parameters the sheets would fail by yielding of the section. The ring compression theory was supported. Their buckling theory, discussed in Chapter 3, indicates that the hydrostatic theory is overly conservative. Meyerhof and  $\Gamma$ 'sher (1963) discussed several field experiences and concluded that failures due to excess deflection were a consequence of unsuitable backfill material or poorly compacted soil. They urged the use of competent backfill so that the ring compression theory could be applied.

# 2.1.6 Large Field Structures

Terzaghi (1943) observed experimental sections of the Chicago subway tunnels in clay, and concluded that a nearly uniform distribution of pressure should be assumed. Terzaghi (1942, p 207) further suggested that the bending moments would be insignificant even in a fairly thick shell because the deformation of the tube automatically reduces the moments.

9

Peck and Peck (1949) discussed observations made on largediameter, flexible steel culverts. They concluded that if the soil is adequately compacted, a moderate deformation will establish a state of nearly uniform all-arcund pressure.

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Lane (1960) described the observation made of tunnel test sections at Garrison Dam. In the flexible sections, the ratio of the horizontal to vertical load ranged from 0.8 to 1.1. However, higher bending moments were observed in the flexible ribs than could be explained by the small differences between the measured horizontal and vertical thrusts. Thus, the moment was apparently dependent on things other than the overall loading, such as the construction procedures.

2.2 Prote tive Structures Research

2.2.1 Dynamic Theory

A number of complex solutions have been generated for mathematical continuum models which are tractable within the classical theory of elasticity. Palmer and Lankford (1963) compared several solutions and recommended the approach taken by Yoshihara and others (1963) as being very promising. Albritton and others (1965) reported the results of an experimental pilot study of a stiff, buried cylinder and an extensive analysis of the mathematical and physical limitations of the currently available continuum theories.

Now (1964) reviewed various dynamic analyses and concluded that "under the assumption of earth media being elastic, homogeneous and isotropic, the dynamic-stress concentration factors for all cylindrical-cavity cases, whether elastically lined or unlined, are all about 10 to 20 percent higher than those for their corresponding static cases." The verification of this analytical prediction could reduce the problem (when a step pulse or instantaneously applied input assumption is applicable) to the simpler static case with an arbitrary 20 percent increase in design equations.

As a consequence of the work of Merritt and Newmark (1962) and Melin and Sutcliffe (1959), Newmark and Haltiwanger (1962) outlined the only theory known to the writer which takes into account the nonelastic behavior of the cylinder.

No directly applicable theory of dynamic buckling is known.

2.2.2 Static Theory

In addition to the mechanics' theories already mentioned in connection with culverts, Section 2.1, several possible elastic continuum theories exist. Palmer and others (1963) compared a number of these and suggested using the solution of Savin (1961) for a lined hole in an infinite plate. Other similar solutions can be found for the static case which evolve as limiting portions (longtime or steady state) of the dynamic analyses where they approach the static case.

2.2.3 Ultimate Strength Laboratory Tests

Bulson (1962) tested 56 thin tubes to failure under static loading up to 100 psi. Overpressure and dial deflections were the only measurements made, but these were sufficient to describe the failure mode as buckling. The failures at the deepest burial, 3/4d, in the dense sand point to a failure mode heretofore unrecognized for fully buried cylinders. Bulson (1963, a and b, and 1965) extended the work to square cylinders and (1964) summarized all of his previous tests.

Donnellan (1964) conducted nondestructive tests on instrumented

cylinders and destructive tests on noninstrumented cylinders buried in dense, dry, 20-30 Ottawa sand. The loading was quasi-static up to a maximum of 160 psi. Only the overpressure was monitored during the ultimate strength tests.

Whitman and Luscher (1962) and Luscher (1963) statically tested small aluminum tubes surrounded by dense sand and symmetrically loaded in a triaxial type device. As a result, Luscher and Höeg (1964) concluded that the major contribution of the sand to the system was to force the cylinder to respond in higher buckling modes. Luscher and Höeg (1964) also conducted buried tube tests which yielded failure conditions similar to those of the fully symmetric situation.

2.2.4 Nondestructive Laboratory Tests

A number of tests have been conducted to verify elastic theories and to form a basis for predicting the ultimate strength of a cylinder.

Allgood and Gill (1964) made a series of static and dynamic tests up to a maximum of 25-psi overpressure on a 24-in.-diameter steel cylinder buried in dense sand. All response was in the elastic range of the cylinder material. They found that the form of the deflection, thrust, and moment distribution was much the same under both types of loading. Some differences were noted: The maximum thrust under dynamic loading was about 14 percent higher than for static loading; the crown deflection under dynamic loading was about twice that under static loading. Allgood (1965), in attempting to summarize the case of a thin metal cylinder buried at shallow depths in a uniform, noncohesive soil, concluded that the net arching (reduction in vertical load below that at the surface) across a thin metal cylinder is negligible.

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Robinson (1962) presented the results of a series of static tests up to a maximum of 100-psi overpressure on 6-in.-diameter tubes buried in dense, dry Ottawa sand. Robinson (1964) extended the earlier tests by including more strain gages. Four test sections were used at a depth of burial of 15 in., 2-1/2d. The results were nonsymmetric in response and showed a great amount of scatter in the moments.

# 2.2.5 Full Scale Tests

Albright and others (1960) described the response of largediameter, buried conduit sections located at the 100-psi pressure range of Shot Priscilla (1957) in Operation PLUMBBOB, a full-scale field test. The sections were selected by means of modified static design procedures, and all survived the blast loading.

Williamson and Huff (1961) described the response of 20-ft long, 7-ft diameter, 10-gauge structural-plate pipes buried at a 10-ft depth of cover and subjected to a pressure of 250 psi from Shot Smoky of Operation PLUMBBOB. Again the structures survived with very small deformations and virtually no damage.

McDonough (1959) described tests on drum-shaped structural models buried at depths of from 0 to 20 ft and subjected to the effects of airinduced pressures resulting from large detonations. The compressibility of the structure relative to the surrounding soil appeared to govern the amount of load that was transmitted to the structure.

### 2.3 Similitude Studies

The American Machine and Foundry Company (1962) and Murphy and Young (1962) examined the feasibility of modeling the soil-structure interaction problem, and developed similitude relations. THE REAL PROPERTY AND A DESCRIPTION OF THE PROPERTY AND A DESCRIPTION OF THE

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Murphy and others (1963) demonstrated the feasibility of using small-scale modeling for qualitative results. Young and Murphy (1964) tested their similitude requirements on stiff aluminum cylinders buried in dry Ottawa sand, and concluded that the requirements were satisfied within the range of parameters investigated.

Dowell (1964) continued the work with stiff cylinders, but experienced difficulty as a result of sidewall friction in the testing device.

## 2.4 Bibliographies and Design Manuals

Van Horn and Tener (1963) and Merkle (1963) prepared annotated bibliographies on the subject of soil-structure interaction. Each chapter of the five volume set of <u>Nuclear Geoplosics</u> by Stanford Research Institute (1964) contains an excellent bibliography. <u>The Effects of Nuclear Weapons</u> by U. S. Atomic Energy Commission (1964) covers the general field of nuclear weapons, and the <u>Proceedings of the Symposium on Soil-Structure</u> <u>Interaction</u> by University of Arizona (1964) presents the most up-to-date research.

Design manuals appeared in 1957 with the U.S. Army Corps of Engineers series EM 1110-345-413 to -421. American Society of Civil Engineers (1961) and Newmark and others (1961) developed design recommendations. Newmark and Haltiwanger (1962, under revision) outlined design procedures for hardened sites.

#### CHAPTER 3. THEORETICAL CONSIDERATIONS

Various theoretical solutions and concepts are presented in this chapter and are compared with the test results in Chapter 6. The nonavailability of a dynamic buckling theory together with the theoretical indication that the dynamic response for a step pulse is only 10 to 20 percent greater than the static response suggests that static theory may be applicable for the elastic case.

### 3.1 Definition of Failure

A protective structure fails when it can no longer perform the function for which it was designed. For the shell under consideration, Fig. 3.1, failure is an inability to keep the ring from collapsing. This could come about by (1) the vertical diameter decreasing to such an extent, say 20 percent, that the crown would reverse curvature and plunge to the invert, Fig. 3.2a; (2) a section of the wall becoming unstable before a large-diameter change has occurred (and buckling inward into the cavity with a large amplitude) as a consequence of the interaction between thrust and moment (a) before any fiber in the cross section has yielded, (b) after some fibers have yielded in bending but before the whole cross section has yielded in thrust, (c) at some time after the whole cross section has yielded in thrust (hoop compression). Fig. 3.2b, c, and d show some observed modes of failure.

Large, i.e. greater than 5 percent, changes in diameter will not occur (if the cylinder is emplaced in a competent backfill) before one of the failure mechanisms in (2) above has triggered the structural collapse. The backfilling around protective structures should be carefully

controlled; therefore, the tests of the present investigation were conducted in well-compacted and -controlled sand and clay specimens.

Because the cylinder tends to readjust itself under load, it may be assumed that the bending moments are negligible in the development of a buckling criterion. Hence, failures (2)(a) and (2)(b) mentioned previously can be considered one condition describing the elastic membrane response of the cross section.

As long as the wall acts as a ductile member, yielding will not constitute failure other than as it precipitates inelastic buckling.

## 3.2 Elastic Buckling

### 3.2.1 Soil Medium Approximated by Water

A first approximation to the problem of a uniform soil-surrounded cylinder can be made by the use of the equation for hydrostatic buckling of a ring, Fig. 3.3. Since this mathematical model assume that the medium possesses no shear strength, it should serve as a lower bound for the buckling value for uniform radial loading. Seely and Smith (1952, p 612) arrived at the classical relation

$$p_{h} = (n^{2} - 1) \frac{EI}{R^{3}}$$
  $n \ge 2$  3.1

where  $p_h = uniform$  collapsing (critical) pressure (force per unit area)

# for the ring section

- n = buckling mode number, an integer
- E = modulus of elasticity of the cylinder material
- I = moment of inertia (per unit length) of the ring cross section
- R = mean radius of the ring

The minimum value for  $p_h$  , other than zero, is

17

$$P_0 = 3 \frac{EI}{R^3}$$

Timoshenko and Gere (1961, p 292) indicated that the buckling forms of higher order can be obtained only by introducing certain additional constraints. For n = 3,  $p_h = 8 \frac{EI}{R^3}$  or 2.7  $p_o$ . For n = 4,  $p_h = 15 \frac{EI}{R^3}$  or 5  $p_o$ . Williamson and Huff (1961, p 42) used 15  $\frac{EI}{R^3}$  as their buckling criterion.

The hydrostatic value for the critical buckling pressure is based on the external forces remaining normal to the surface of the ring when buckling occurs. Boresi (1955, p 101) has shown that the coefficient on  $\frac{\text{EI}}{\text{R}^3}$  in equation 3.2 is 4.5 for the fundamental buckling mode if the external forces are assumed to remain directed toward the original center of the ring instead of normal to the surface. Bodner (1958) showed that the coefficient is 4 for a constant-directional-pressure force system.

The foregoing observations indicate some of the potential weaknesses in the hydrostatic assumption. A slightly different assumption in the action of the surface traction could change the critical buckling pressure by 50 percent.

Anderson and Boresi (1962) investigated a nonuniform load distribution of the form  $p = p_a \sin^2 \theta$ , Fig. 3.4, where  $p_a$  is the peak pressure at the crown. For centrally directed forces,  $p_{cr}$  (average) = 4.5  $\frac{EI}{R^3}$ , which was identical with the uniform load case where  $p_{cr}$  (average) is the total load divided by the circumference. This implies that the specific load distribution may not be overly critical in some cases.

For the test specimens of cylinder groups A, B, D, and E,  $p_0 = 135$ psi and for group C,  $p_0 = 5.1$  psi from equation 3.2 for the lowest mode.

Other investigators, e.g. Donnellan (1964), have tested cylindrical shells in which the longitudinal boundaries were supported and as a result the theoretical buckling equation became a function of the cylinder length, *i*. Timoshenko and Gere (1962, p 478) derived the expression for a simply supported shell,  $w = \frac{\partial^2 w}{\partial x^2} = 0$  where w is the deflection of the middle surface in the radial direction and x is the cylinder coordinate in the longitudinal direction, Fig. 3.1.

$$\mathbf{p}_{t} = \frac{\mathbf{E}\mathbf{h}}{\mathbf{R}(\mathbf{n}^{2}-1)\left(1+\frac{\mathbf{n}^{2}\boldsymbol{s}^{2}}{\boldsymbol{x}^{2}\mathbf{R}^{2}}\right)} + \frac{\mathbf{E}\mathbf{I}}{(1-\boldsymbol{v}^{2})\mathbf{R}^{3}}\left(\mathbf{n}^{2}-1+\frac{2\mathbf{n}^{2}-1-\boldsymbol{v}}{1+\frac{\mathbf{n}^{2}\boldsymbol{s}^{2}}{\boldsymbol{x}^{2}\mathbf{R}^{2}}}\right) \qquad 3.3$$

where  $p_t$  is the theoretical buckling pressure, and h is the wall thickness. The number of half-waves, n, into which the shell buckles increases as the length of the shell decreases and as the thickness of the shell decreases. Taking the limit of equation 3.3 as the length becomes long (approaches infinity) yields the equation for a long tube or structure

$$P_t = \frac{(n^2 - 1)}{(1 - v^2) R^3} \frac{EI}{R^3}$$

3.4

where v is Poisson's ratio of the cylinder material.

For a value of v = 0.3, equation 3.4 for a long cylindrical shell differs from equation 3.1 for a ring by only 10 percent.

Armenakas and Herrmann (1963) reanalyzed the shell case and presented convenient graphs to allow rapid assessment of the critical buckling number n corresponding to values of S/R.

3.2.2 Soil Medium Approximated by Elastic Support

Cheney (1963, p 41) derived an expression for the critical buckling pressure  $(p_c)$  of a ring with radial elastic support, Fig. 3.5.

$$p_c = (n^2 - 1) \frac{EI}{R^3} + \frac{k_z R}{n^2 - 1}$$
  $n \ge 2$  3.5

in which

$$n_{\rm cr} = \sqrt{1 + \sqrt{\frac{k_z R^4}{EI}}} \ge 2$$
 3.6

This leads to a convenient approximation

$$\mathbf{p}_{c} = 2\sqrt{k_{z}\frac{EI}{R^{2}}}$$
 3.7

where  $k_z$  is the spring constant in psi per in. of radial deflection. Cheney (1964) pointed out that equation 3.7 underestimates the buckling load no more than by 10 percent for n greater than 5 and less than 1 percent for n greater than 10. For vanishing values of  $k_z$  and for n less than 5, the exact expression, equation 3.5, must be used because equation 3.7 is not suited to small values of the spring constant or n.

The great difficulty involved in applying this type of equation is the evaluation of an appropriate spring constant,  $k_z$ , for the soil. To facilitate comparison, equation 3.7 can be rewritten as

$$p_{c} = 2\sqrt{k_{z}R^{4}}\sqrt{\frac{EI}{R^{3}}}$$
3.8

Meyerhof and Baikie (1963, p 13) arrived at an elastic buckling equation by modifying the theory of flat plates on an elastic foundation. Their equation may be written as

$$P_{\rm m} = \frac{(n+1)^2 - 1}{1 - v^2} \frac{\rm EI}{\rm R^3} + \frac{(1 - v^2) \, \rm k_{\rm m} \rm R}{(n+1)^2 - 1} 3.9$$

where k is the coefficient of soil reaction ("subgrade modulus").

20

For large values of n this can be reduced to

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$$p_{\rm m} = 2 \sqrt{\frac{k_{\rm m}^{\rm EI}}{(1 - v^2)R^2}}$$
 3.10

or

$$p_{m} = 2 \sqrt{\frac{k_{m}R}{(1 - v^{2})}} \sqrt{\frac{EI}{R^{3}}}$$
 3.11

Equation 3.11 differs from Cheney's equation, 3.8, only by the factor  $(1 - v^2)$ .

Luscher and Höeg (1964, p 35) used an approach of Hetényi (1946) to arrive at an equation for critical buckling pressure  $(p_{\ell})$ .

$$p_{l} = 2\left(\sqrt{\frac{k_{l}R^{3}}{EI} + 1} - 1\right)\frac{EI}{R^{3}}$$
 3.12

where

$$n_{c.r} = \sqrt[4]{\frac{k_{\ell}R^3}{EI} + 1}$$
 3.13

These can be simplified for higher order buckling modes to

$$\mathbf{p}_{\mathbf{f}} = 2\sqrt{\mathbf{k}_{\mathbf{f}}} \frac{\mathbf{EI}}{\mathbf{R}^3} = 2\sqrt{\mathbf{k}_{\mathbf{f}}} \sqrt{\frac{\mathbf{EI}}{\mathbf{R}^3}} \qquad 3.14$$

and

$$n_{cr} = \sqrt[4]{k_{\ell} \frac{R^3}{EI}} 3.15$$

where  $k_g = \text{coefficient}$  of elastic soil reaction (having the units psi per strain). Luscher and Höeg (1964, p 143) expressed  $k_g$  in terms of the constrained tangent modulus of the soil and the thickness of the soil support. For the Ottawa sand which they used, the equation was written as

$$P_{R} = 780 \left[ \frac{EI}{R^{3}} f(R) \right]^{5/6}$$
 3.16

where f(R) is a function of the depth of burial.

Newmark and Merritt (1963) considered a similar problem.

All of the above can be summarized by the following:

$$p_c = 2 \sqrt{k_z R} \sqrt{\frac{EI}{R^3}} \ge \frac{3EI}{R^3}$$
 3.8

$$p_{\rm m} = 2 \sqrt{\frac{k_{\rm m}R}{(1-v^2)}} \sqrt{\frac{EI}{R^3}}$$
 3.11

$$\mathbf{p}_{\boldsymbol{\ell}} = 2\sqrt{k_{\boldsymbol{\ell}}} \qquad \sqrt{\frac{\mathbf{EI}}{\mathbf{R}^3}} \qquad 3.14$$

The application of this type of formula revolves around an ability to arrive at an appropriate value of the coefficient of soil reaction. This will be discussed in Chapter 6.

## 3.2.3 Soil Medium Approximated by an Elastic Medium

Forrestal and Herrmann (1964) derived a buckling equation for a long cylindrical shell subjected to uniform external pressure exerted by a surrounding elastic medium, Fig. 3.6. The solution for the unbonded case (shear stresses between the shell and the medium are absent) can be expressed as

$$P_{f} = \frac{(n^{2} - 1)}{(1 - v^{2})} \frac{EI}{R^{3}} + \frac{E_{g}}{(1 + v_{g})(1 - 2v_{g})(n + 1) + n} \qquad 3.17$$

where  $p_{f}$  is the critical buckling pressure,  $E_{g}$  is the Young's modulus of the medium, and  $v_{g}$  is the Poisson's ratio of the medium. Solutions for the bonded case were also presented but were more complicated and did not give results which varied greatly from those for the unbonded case.

### 3.3 Inelastic Action

After the cross section has yielded in hoop compression, it can

continue to yield or strain for some time before structural collapse. It is hypothesized that such failure can be defined by the judicious choice of a ductility factor. Newmark and Haltiwanger (1962) defined this factor,  $\mu$ , as the ratio of the maximum deflection to the deflection at yield. Ductility factors for compression members have been assumed to be in the range 1.3 to 1.5.

#### 3.4 Characteristic Ring Parameter

In order to compare the results of various tests run by different investigators, it is necessary to have a parameter by which the ring can be adequately described. Various groupings have been used, e.g. radius to thickness ratio, diameter to thickness ratio, and these quantities weighted in some fashion by the modulus of elasticity.

The quantity  $\frac{EI}{R^3}$  appears as a parameter in all of the aforementioned buckling equations and appears to be a convenient index for the elastic action of rings.

Stiffness can be defined as the force required to produce a unit deflection. For a large variety of loading configurations this is a function of  $\frac{\text{EI}}{\text{R}^3}$ . Fig. 3.7 illustrates a number of these loading conditions, many of which were investigated by Lane (1960, p 287). Point load, P (Fig. 3.7a):

$$\frac{P}{\Delta_{v}} = 6.7 \frac{BI}{R^{3}}$$
3.18

60° triangle (Fig. 3.7b):

$$\frac{p(2R)}{\Delta_v} = 29 \frac{E1}{R^3}$$
 3.19

90° triangle (Fig. 3.7c):

$$\frac{p(2R)}{\Delta_{v}} = 22 \frac{BI}{R^{3}}$$
 3.20

120° triangle (Fig. 3.7d):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 19 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.21

180° triangle (Fig. 3.7e):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 18 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.22

Parabolic (Fig. 3.7f):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 14 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.23

Uniform (Fig. 3.7g):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 12 \frac{\mathbf{EI}}{\mathbf{R}^3} \qquad 3.24$$

Side support (Fig. 3.7h):

$$\frac{p(2R)}{\Delta_v} = 12 (1 - q) \frac{EI}{R^3}$$
 3.25

Uniform radial (Fig. 3.)a):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 2 \frac{\mathbf{E}\mathbf{h}}{\mathbf{R}} \qquad 3.26$$

where  $\Delta_{y}$  is the decrease in vertical diameter, q is the ratio of the horizontal to the vertical pressure, and h is the ring wall thickness.

It also appears that the parameter  $\frac{EI}{R^3}$  may provide a means for differentiating between so-called stiff and flexible buried cylinders. The lowa Formula (Fig. 2.1c) can be rewritten as

$$\frac{p(2R)}{\Delta_{h}} = \frac{0.061(k_{B}R) + \frac{SI}{R^{3}}}{0.083}$$
 3.27

where  $\Delta_{\rm h}$  is the increase in horizontal diameter. If a flexible structure is defined as one whose stiffness,  $\frac{\rm HI}{\rm R^3}$ , has less than a 10 percent influence on elastic deformation relative to the influence of the soil, then, from equation 3.27, a stiff structure is one in which

$$\frac{EI}{R^3} > 0.61(k_sR)$$
 3.20

In a dense sand medium (with  $k_{s}R = E' = 1000$  as suggested by Watkins and Nielson (1964, p 173)), a cylinder is stiff if  $\frac{EI}{R^3} > 610$  psi from equation 3.28. In a clay (with E' = 900), it is stiff if  $\frac{EI}{R^3} > 550$ psi. These stiffness values are greater than those required to prevent buckling for overpressures lower than 1500 psi.

Other approaches have been suggested to arrive at relative stiffness. Meyerhof and Baikie (1963) indicated that the relative stiffness, S , of a culvert with respect to the soil is

$$S = \sqrt[4]{\frac{EI}{(1 - v^2)k_m}} 3.29$$

 $\mathbf{or}$ 

$$S = \sqrt{\frac{3}{2(1 - v_s^2)EI}}{(1 - v^2)E_s}$$
 3.30

where  $v_s$  is Poisson's ratio of the soil. Davisson\* suggested that relative stiffness,  $S_1$ , could be expressed as

$$S_{1} = \sqrt[3]{\frac{EI}{RE_{s}}} \qquad 3.31$$

and that a typical discriptor, TD , would be

$$ID = \frac{R}{S_1}$$
 3.32

No numerical limits have been established to differentiate between stiff and flexible structures on the basis of these equations.

<sup>\*</sup> Private communication with M. T. Davisson, Professor, Department of Civil Engineering, University of Illinois, June 1964.

Qualitatively, a flexible structure may be thought of as one which deforms (vertical change or volumetric change) more than the medium replaced would have. However, this concept has its greatest applicability in the assessment of overall arching.

Flexibility, in the structural sense that it will deform sufficiently to mobilize the passive resistance of the side-supporting soil, appears to be assured for a structure made of ductile material whose value of  $\frac{\text{EI}}{\text{R}^3}$  is less than about 600 psi.

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#### CHAPTER 4. EXPERIMENTAL PROCEDURE

## 4.1 Description of Cylinders

### 4.1.1 Considerations in Selection of Design

A number of practical considerations were influential in the selection of the cylinder material and the geometric dimensions for the tests.

Aluminum was selected for the cylinder material because it, in general, is not strain-rate sensitive according to Steidel and Makerov (1960) and Smith (1963). It has a face-centered, cubic, crystalline, lattice structure and exhibits a continuous stress-strain curve with no sharp yielding zone. Steel was rejected because of its unpredictable yield strength under dynamic loading. Massard and Collins (1958) and Wright and Hall (1964) have proposed methods of taking this strain-rate effect into account, but it was considered best to avoid adding this parameter to the study. Plastics are made of long chain molecules which possess no ordered geometric pattern of structure, and hence are not only strain-rate sensitive but also experience a brittle failure under rapid loading as indicated by Dietz and McGarry (1956) and Hall (1958).

The relative size of the cylinders was dictated by the dimensions of the University of Illinois 2-ft-diameter, 500-psi, loading device. As a result, it can be assumed that for shallow burial no load was lost due to the effect of sidewall friction, and hence that the free-field vertical soil pressure immediately above the cylinder was equal to the surface overpressure. Measurements by Hanley (1963) have shown this to be a reasonable assumption.

The specific cross-sectional dimensions were determined by consideration of two factors. First, it was essential to have specimens that would fail under the maximum available pressure. In this regard, it was also desirable to take full advantage of the high pressure capability available by concentrating on specimens which would be too strong for ultimate strength studies in other facilities. Second, in view of the high cost of specimen preparation and the desirability of testing a large number of cylinders, commercially available tubing was sought.

The length was governed by the desire to have a somewhat realistic proportion between length and diameter, and by the need for enough length to smooth out any local disturbances caused by the presence of either the outside strain gages or end walls. Also, the length should be long enough to allow two-dimensional behavior and short enough to fit conveniently into the tank.

The closure plates (end caps) for the ends of the cylinder were designed so that no axial loading would be transferred to the cylinder, while at the same time retaining free radial motion.

4.1.2 Cylinder Material

Although all of the cylinders are made of aluminum, alloys with three different, nominal yield strengths were involved. The stress-strain properties of the materials were experimentally obtained and are discussed in Appendix A. The modulus of elasticity, E, was found to be a constant value,  $10 \times 10^6$  psi. Two yield values were determined: a lower yield point,  $\sigma_{y1}$  (which is hard to define and probably no more accurate than ±10 percent), corresponding to the first noticeable deviation from elastic behavior; and an upper yield point,  $\sigma_{y2}$ , corresponding to the

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stress which would result in 0.2 percent permanent strain. These values are summarized in Table 4.1.

4.1.3 Cylinder Geometry

The outside diameter, d , for all cylinders was 3.5 in. Micrometer measurements of the horizontal and vertical diameters prior to each test indicated that the greatest deviation was  $\pm 0.5$  percent. The larger diameter was oriented vertically for the test. The length,  $\ell$ , was a constant 10.5 in., making the length-to-diameter ratio for all cases equal to 3. Two wall thicknesses were used, 0.065 in. and 0.022 in. No deviation in thickness was found to be greater than  $\pm 0.001$  in. A longitudinal section of a cylinder is shown in Fig. 4.1, and the geometric values are summarized in Table 4.1.

4.1.4 End Conditions

The conditions at the ends of the cylinder represent a free boundary. The end caps prevented the transfer of any axial load to the cylinder and the clearance of 0.05 in. at each end was sufficient to allow for radial motion. One layer of commercial, paper masking tape was used to hold the cylinder in place between the end caps during handling and placement in the soil.

4.1.5 Natural Period of Vibration

In dynamic problems it is sometimes necessary to know the natural period of vibration of the structure for all loading conditions except a step pulse. For circular, cylindrical structures buried underground the procedure for determining the period is not well established. However, a good approximation can be made by finding the period of a cylinder in air and making appropriate corrections to account for the soil.

The natural period of the pure radial vibration of a complete

ring is given by Timoshenko and Young (1955, p 426) as

$$T_{c} = 2\pi \sqrt{\frac{\gamma R^{2}}{Eg}}$$
 4.1

where

 $T_c = natural compressive period$ 

 $\gamma$  = specific weight

R = radius of the center line of the ring

E = modulus of elasticity

g = acceleration due to gravity

For this study  $\gamma = 169 \text{ lb/ft}^3$ ,  $E = 10 \times 10^6 \text{ psi}$ ,  $g = 32.2 \text{ ft/sec}^2$ , and R = 1.72 in. (groups A, B, D, E) or 1.74 in. (group C). The calculations yield for all cylinders

$$T_{2} = 0.06 \text{ msec}$$
 4.2

For comparison, consider the period of the fundamental mode of flexural vibration given by Timoshenko and Young (1955, p 429) as

$$\Gamma_{f} = 2\pi \sqrt{\frac{5}{36}} \frac{\gamma A R^{4}}{EgI}$$
 4.3

where

Tf = natural flexural period
A = area of the cross section perpendicular to the ring
center line

I = moment of inertia of the cross section perpendicular to the ring center line

This may be rewritten as

$$T_{f} = \frac{R}{h} \sqrt{\frac{5}{3}} 2\pi \sqrt{\frac{\gamma R^{2}}{Eg}}$$
 4.4

where h = thickness of the ring.

29

The substitution of equation 4.1 into equation 4.4 yields

$$T_{f} = \frac{R}{h} \sqrt{\frac{5}{3}} T_{c}$$
 4.5

For this study h = 0.065 in. (groups A, B, D, E) or 0.022 in. (group C). The calculations yield

$$T_f = 1.9 \text{ msec}, \text{ groups A}, B, D, \text{ and E}$$
 $T_r = 5.6 \text{ msec}, \text{ group C}$ 
4.6

The soil acts in two ways to modify the foregoing expressions for the natural period. It tends to stiffen, and at the same time to add mass to the structure. The effect of the mass of soil, virtual mass, which must be accelerated along with the buried structural elements can be treated in the manner suggested by Merritt and Newmark (1964, p 23); but, the deflections observed in this study for the small cylinders were of such small magnitude that it is unlikely that any appreciable amount of additional mass should be included. The stiffening effect is even less susceptible to quantitative assessment.

4.2 Description of Soil

### 4.2.1 Considerations in Selection of Test Soils

Although considerable thought is being given to what soil parameters govern soil-structure interaction, no complete answer is presently available. Therefore, it was desirable to use soils at each end of the spectrum,\* and at the same time soils whose shear strength and

<sup>\* 1</sup>st Lt. A. J. Hendron, Jr., Ph.D., "A Short Technical Note on the Extremes in Soil Types in Regard to Dynamic Soil-Structure Interaction," Vicksburg, Miss., July 22, 1964.

stress-strain properties could be documented for future reference. A new soil environment was built for every cylinder; hence, the in-place properties of the soils used had to be reproducible. Dense, dry sand and a clay of high plasticity were selected. The sand was uniformly graded because a given density was thought to be more reproducible in a uniformly graded sand than in a well-graded sand.

### 4.2.2 Sangamon River and Cook's Bayou No. 1 Sands

The Sangamon River sand has been used extensively in tests at the University of Illinois. It was used in a dense  $(D_r = 78\%)$ , dry condition as the soil environment for the testing of cylinder groups A, B, and C. The Cook's Bayou No. 1 sand  $(D_r = 79\%)$  has been used for several experiments at WES; extensive, dynamic one-dimensional and triaxial tests are planned in the near future to expand the knowledge of its properties. It was used for group E. The characteristics of both sands, together with the placement techniques employed, are outlined in Appendix B.

4.2.3 Buckshot Clay

This particular clay (CH) was selected for the group D cylinders because of the experience at WES in its use. However, even with this kind of knowledge available, great difficulty was experienced in developing placing methods adaptable to this study. The properties and placement techniques are discussed in Appendix C.

4.3 Loading Devices

Experimental work in this area has required the development of new testing machines.

4.3.1 Illinois

The equipment used in the first stage of this study was

31

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originally developed by Egger (1957) and later modified to permit simulation of blast loading by Sinnamon and others (1961). Its capabilities are described by Sinnamon and Newmark (1961), and it has recently been used by Hanley (1963) to study the interaction between sand and vertically oriented cylinders.

The container is a vertical cylinder 26-3/4 in. high and 23-1/4in. in diameter. A 1/32-in.-thick neoprene diaphragm is placed over the soil surface to prevent gas penetration. Then a spacer ring is positioned, followed by the static or dynamic loading head. The device is illustrated in Fig. 4.2. Both the static and dynamic loads are provided by a compressed gas system. Although the equipment is capable of producing rise times in the neighborhood of 3 msec by using helium gas, this study was conducted with nitrogen gas because it is less expensive and because the 3 msec rise time apparently offered little advantage over the 13 msec rise time (rapid) with nitrogen gas. A typical overpressure-time relation is shown in Fig. 4.3. No reflection of the incident wave on the bottom was noted.

4.3.2 WES

Cylinder groups D and E were tested in the Small Blast Load Generator (SBLG) facility at WES. This was the first extensive experimental program completed in the SBLG and hence a number of problems in technique had to be resolved during the course of the investigation. The dynamic overpressure is applied by the detonation of two parallel lines of PETN in the form of primacord. The effective overpressure-time relation (dynamic) is shown in Fig. 4.3. The early part of the curve was obtained by averaging the maximum and minimum points in adjacent oscillations.

Although the amplitude of the oscillations varied as much as  $\pm 50$  percent from the average, the impulse was so small (10,000 and 20,000 cps ringing) that the approximation in Fig. 4.3 is justified. The high-frequency signals were probably caused by the nonshock isolated gage mounts. The pressure distribution on the surface is within  $\pm 10$  percent of being a plane wave according to Kennedy and Sadler (1965).

The loader is a cylindrical ring device, 46-3/4 in. in diameter. For these tests an average soil replacement depth of 2 ft was used. The layout is shown in Fig. 4.4. The static tests of group D were run with a rigid concrete base (III). The static tests of group E along with the dynamic tests of both D and E were conducted with a pseudo-infinite base (II) to avoid the dynamic disadvantages of the rigid base.

The "infinite" base is a column of sand extending 9 ft below the floor level. This column had been previously loaded many times to 500 psi, and no further compaction was observed. Two feet of sand above floor level was replaced for each sand test. For the dynamic clay tests, a rubber diaphragm was inserted at floor level to separate the lower sand column from the upper 2 ft of clay.

The operation of the loading device has been outlined by Boynton Associates (1960), and the U. S. Army Engineer Waterways Experiment Station (1963) and an evaluation study is being made by Kennedy and Sadler (1965).

# 4.4 Instrumentation

### 4.4.1 General

Letal film strain gages were used to measure hoop strain on the inside and outside of the cylinders (Fig. 4.1). Static deflection gages were made from brass shim stock and individually calibrated. The transducers and techniques are discussed in Appendix D.

4.4.2 Illinois

The instrumentation used is pictured in Fig. 4.5. The active strain gage on the cylinder was one arm of a four-arm bridge. The dummy gages were mounted on isolated metal strips outside the test tank. Multiconductor cable was used initially, but it was found that two-conductor shielded cable provided a better barrier to spurious noise in the system.

The eight hoop strain gages were hooked to a bank of Consolidated Electrodynamics Corporation (CEC) carrier amplifiers, Type 1-127. A 12-channel CEC, direct-write, recording oscillograph Type 5-124 with available paper speeds of 0.5, 2, 8, 32, and 128 in./sec was used. The two deflection gages each formed two arms of a bridge and were fed through DANA d-c amplifiers to the oscillograph. For the static tests, the slowest paper speed was used. A timing trace of 2 cps and one reference (dead) trace completely utilized all of the available channels. The overpressure was read on an auxiliary Bourdon gage with the timing trace interrupted at predesignated pressure levels. Modifications were made for the rapid tests. The output of the strain gage amplifiers was split so that it was placed on both the oscillograph and a Honeywell 8100 tape recorder (as a back-up record). Additional DANA amplifiers were used to drive the tapes. The time base frequency was increased to 500 cps. The output of a Kistler Instrument Corporation, piezoelectric, pressure transducer, which was in series with a Kistler calibrator and charge amplifier, was used to record pressure. The recording paper was driven at the fastest speed possible, 128 in./sec.

The frequency response of the oscillograph system was limited to

that of the CEC 7-364 galvenometers, 500 cps. The tape system had a frequency response of at least 3000 cps and a few records reproduced directly from the tape indicated that no frequencies higher than 500 cps were present.

## 4.4.3 WES

The equipment used for group E (the first test series at WES) and the evaluation of the overpressure-time signature is shown in Fig. 4.6. The Wheatstone bridge was set up as in the Illinois tests. The Sensor Analog Module (SAM) amplifiers used are d-c, and hence the dynamic frequency response was again limited by the galvanometer capabilities, 2500 cps (CEC 7-362).

After the group E tests were completed, the SBLG facility instrumentation was moved to a separate area. The layout used for the group D tests is shown in Fig. 4.7. In this case, DANA amplifiers coupled with galvo drivers were used.

Overpressure was monitored by a pair of 1000-psi Norwood pressure transdulers, Model 211C. Additional pressure transducers were used and their output recorded on tape to gain higher frequency response (20,000 cps) in order to describe adequately the high-frequency characteristics of the pressure-time signature.

4.4.4 Sources of Error

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Fotential sources of error are present throughout the system: (1) inexact strain gage placement  $(\pm 2 \$)$ ; (2) variation in gage factor and resistance  $(\pm 1 \$)$ ; (3) amplifier nonlinearity  $(\pm 2 \$)$ : (4) galvanometer nonlinearity  $(\pm 1 \$)$ ; and (5) properties of the pressure transducers  $(\pm 5 \$)$ . These imply a confidence limit of no better than  $\pm 11$  percent in the instrumentation system.

35

CHAPTER 5. ESENTATION OF EXPERIMENTAL RESULTS

## 5.1 Method of Presentation

# 5.1.1 Cylinder Coding

Table 5.1 outlines the overall testing program for the 46 specimens and identifies each cylinder with its respective soil environment, depth of burial, and type of loading. The notation used, e.g. A-3, to identify each cylinder (and thus each test) has general meaning. The alphabetic term, A, was used to identify the original 12-ft tube from which the test cylinder was cut and can be related to the stress-strain curves of Appendix A. Cylinders with a numerical designation 1 through 5 were tested statically, while those designated 6 through 10 were tested either rapidly or dynamically. In Tables 5.2 through 5.11 the tests are presented by group (A, B, C, D, E), static first, in the order of increasing depth of burial within the group.

### 5.1.2 Tables of Data

The digitized strain values were taken from the oscillograph records at points corresponding to specific values of the overpressure to obtain a cause-and-effect relation. In the dynamic tests, peak strain values were recorded. These experimental strain values, together with diameterchange values (for static tests only), are listed in Tables 5.2 to 5.11 with respect to overpressure.

Use of a dash instead of a number indicates that the results were lost due to instrumentation difficulties. The values of stress, threat, and moment are also listed in the tables. The gage locations are identified in Fig. 4.1.

36

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## 5.1.3 Data Plots

The values of strain were, in general, not plotted directly in Figs. 5.1 to 5.43 because an appropriate scale to show the large inelastic values would have masked the much smaller elastic strains. The stress to cause yield and the thrust to cause yield are shown by horizontal dotted lines in each figure. "First yield"  $(\sigma_{y1})$  represents the stress at a point where the slope of the stress-strain curve departs from the initial elastic slope (E). The yield value corresponding to 0.2 percent permanent strain is the "0.2 percent offset yield"  $(\sigma_{y2})$ . The diagonal dotted line labeled "uniform radial load" represents the theoretical relation derived for a uniform radial load equivalent in magnitude to the overpressure, Fig. 2.1d.

Stress, thrust, moment, and diameter change (static tests only) are plotted as ordinates with respect to the surface overpressure as the abscissa.

The symbols used to identify a gage location are presented on each figure and are consistent throughout. The inside gages are represented by open symbols and the outside gages by closed symbols. The cross sections are identified by the applicable open symbol.

#### 5.2 Computations

#### 5.2.1 Moment and Thrust Computation

The moment and thrust at a cross section were calculated from



37

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where  $M_y$  is the moment in the y or tangential direction, Fig. 3.1, in units of pounds (inch-pounds per inch), and  $N_y$  is the thrust or in-plane force in the tangential direction in units of pounds per inch. For the elastic case these can be reduced to

$$M_{y} = (\epsilon_{e} - \epsilon_{i}) \frac{Eh^{2}}{12}$$
 5.3

$$N_{y} = (\epsilon_{e} + \epsilon_{i}) \frac{Eh}{2} \qquad 5.4$$

where  $\epsilon_e$  is the exterior strain, and  $\epsilon_i$  is the interior strain at the cross section in inches per inch. Compressive strains and thrust are considered positive in the presentation. Moment tending to compress the external fibers is positive.

## 5.2.2 Computer Program

To reduce the large mass of strain data to applicable stress, thrust, and moment values, a program (13-G1-Z5010) was written in FORTRAN for the WES, GE 225 computer. The aluminum stress-strain curves of Appendix A were input in a discrete number of linear segments and a "table lookup was utilized to compute the elastic and inelastic stress. The strain distribution was assumed to be linear across the section, Singer (1951, p 409), so that the expressions for moment and thrust, equations 5.1 and 5.2, could be numerically integrated for the nonelastic case. The program assumes that the material stress-strain properties are the same in both tension and compression and that any unloading takes place along the original load curve.

## 5.2.3 Computation of q

Values of q are listed in Tables 5.2 to 5.11. As used in this

context, q is not a coefficient of earth pressure, but merely defines the atio of the average thrust at the crown and invert divided by the average thrust at the spring line. Values of q are plotted in Figs. 6.1 to 6.3. 5.3 Mode of Failure

All of the cylinders that failed, failed by a catastrophic snapthrough (caving) of the crown. A noise was heard at the moment of failure and all of the strain gage traces were instantaneously driven off the oscillograph, either by being overranged or by shorting out electrically. The last recorded strains in the tables are those at the moment of failure.

The failed cylinders are shown in Figs. 5.44 and 5.45. The distorted cross section of two cylinders which did not fail are shown in Fig. 5.46 (the strain gage wires are evident in D-6), and the postfailure clay configuration is illustrated in Fig. 5.47. A plot of overpressure at failure versus depth of burial is shown in Fig. 5.48.

5.4 Stress, Moment, and Thrust

The cylinder groups are presented in the order A, B, C, E, and D because the first four groups were in a sand medium and the last in clay.

5.4.1 A Group

The static test data are presented in Table 5.2 and plotted in Figs. 5.1 through 5.6. An air line broke at 400 psi during test A-3. Fig. 5.4, test A-3A, presents the data up to that point. The line was repaired, the gages were rezeroed, and a second test, A-3B, Fig. 5.5, was run up to 500 psi. The values of stress, thrust, and moment listed for test A-3B were computed by the computer program on the assumption of no residual strain. Sample calculations based on the more realistic assumption of residual strains from test A-3A indicated that the listed values are no more than about 10 percent low.

The deflection gages were not suitable for the rapid testing, and hence data from them do not appear in Table 5.3 nor in Figs. 5.7 through 5.11.

5.4.2 B Group

The static test data are presented in Table 5.4 and plotted in Figs. 5.12 through 5.17. The B group was the first group to be tested, and B-1 was the first cylinder. Test B-1A, Fig. 5.12, terminated at 300 psi because no higher pressure was attainable with the loading device. A subsequent modification in the O-ring configuration allowed the device to attain its 500-psi static capacity. Test B-1 was rerun, test B-1B, Fig. 5.13, and the cylinder failed at 315-psi overpressure.

The rapid test data are presented in Table 5.5 and plotted in Figs. 5.18 through 5.22.

5.4.3 C Group

The static test data are presented in Table 5.6 and plotted in Figs. 5.23 through 5.27. The rapid test data are presented in Table 5.7 and plotted in Figs. 5.28 through 5.32.

5.4.4 E Group

The static tests were run as duplicates to check the tests of the A group. Test data are presented in Table 5.8 and plotted in Figs. 5.33 through 5.35. The dynamic results (peak strain values) are presented in Table 5.9 and plotted in Figs. 5.36 and 5.37. The initial pressure rise of the dynamic pressure wave, Fig. 4.3, approximates a step pulse. For this region a strain-pressure relation is unmanageable. Therefore, the dynamic results are plotted with respect to the circular angle  $\theta$  (Fig. 4.1) for the various overpressures attained. No failures resulted from the maximum available, nominal overpressure of 250 psi.

5.4.5 D Group (Clay)

The static test data are presented in Table 5.10 and plotted in Figs. 5.38 through 5.42. The dynamic results are presented in Table 5.11 and plotted with respect to the circular angle  $\theta$  in Fig. 5.43. The values of stress, thrust, and moment were computed by the computer program on the assumption of no initial strain. Sample calculations, which took into account the strains impressed during placement, indicated that the values listed in Tables 5.10 and 5.11 are no more than about 10 percent low.

41

### CHAPTER 6. ANALYSIS AND INTERPRETATION OF TEST RESULTS

Initially this discussion will concern Figs. 5.1 through 5.48; then other detailed comparisons of pertinent aspects of the data will be treated.

6.1 Overall Structural Response

6.1.1 A Group (Sangamon Sand)

Fig. 5.1, test A-1 (Z = 0 in.), depicts the structural response of a relatively stiff cylinder as it progresses toward failure under static loading. This is a typical case only for cylinders buried at depths approaching zero depth of burial. It is evident that the stress curves are not linear functions of the applied pressure even in the elastic range of the cylinder material; the lower stresses (those tending to tension) are the ones most susceptible to nonlinear behavior. The agreement of the stress levels for gages 2 and 4, and 2a and 4a indicates that the cylinder experienced generally symmetric response about the vertical axis. The crown and invert at this very shallow burial did not exhibit this agreement in response. The stress at many gage points was greater than the first yield stress of the cylinder material. Only the stress recorded for the outside gage at the crown, la , tended to pass the 0.2 percent offset yield stress of the material (at incipient failure).

Thrust is a more nearly linear function of overpressure than the stress at any gage point. The thrusts at the four cross sections are nearly equal below 150 psi; but at high pressures the thrust at the invert is considerably lower than the thrust at the crown or spring line for the case of shallow burial.

The decidedly nonlinear variation of moment with overpressure above 100 psi is the consequence of the cylinder readjusting itself under load and probably of the load distribution changing. It is important to note how the magnitude of the spring line moment decreases for input pressure greater than 200 psi. It is at this pressure that the stresses exceeded the first yield stress of the structural material. The change in sign of the crown moment is of concern. For the structure to assume an elliptical shape (with the major axis horizontal), it would seem that the crown moment would have to be positive throughout the loading. However, this is not the case for pressure levels below 210 psi. Coupled with this, the diameter changes are extremely small for the first 210 psi of loading. This type of reversal of curvature at the crown was not an isolated occurrence. It is shown in the results of test A-5 in Fig. 5.2 and in other cylinders which are very close to the surface boundary and susceptible to collapse. There are a number of possible explanations for this phenomenon.

- The vertical axis was slightly greater than the horizontal axis, and this by itself may have influenced the sign of the moment prior to incipient failure. However, if this were significant it would have influenced the moments at deeper depths of burial.
- 2. The external strain gages and their respective protective covering could cause load concentrations away from the gage locations by activating local arching. But, this would not be the case with the depth of burial, Z, equal to zero.

- 3. Nonuniformity in the soil medium could cause uneven stress distribution. Again, this should be a random occurrence, while the phenomenon is systematic.
- 4. The tendency to buckle in a mode other than the lowest mode could cause local moment anomalies. Higher order buckling modes would have node points occurring in a random fashion even though collapse came by a full snap-through (caving) of the crown. But, here too the occurrence would be random.
- 5. The proximity of the crown to the surface boundary at very shallow burial, relative to the proximity of other points, is much more significant than at deeper depths of burial. The load at the crown is fixed, but local arching could have caused an uneven load distribution. At the deeper depths enough soil would be present to smooth out the local variations.

DaDeppo (1963, p 30) concluded that the magnitude of initial deformation in arches was important in controlling the flexural response. He was most concerned with variation in the initial shape induced by backfilling. However, the conclusion would apply regardless of how the variations in initial shape came about. Random deviations of the cylinder from circularity could result in random moment response. But, the moment response in the present investigation was systematic and repeatable.

Robinson (1964) recorded moments on a cylinder at every 45-degree point, and they were all of the same sign. He felt that this was due to local arching of the soil at the contact between the external strain gages

and the soil. However, the data were not reproducible.

It is the writer's opinion that the most plausible explanation of the negative moment is directly related to the proximity of the surface boundary causing local arching to neighboring elements of the cylinder. The buildup in pressure and subsequent nonuniform loading become less significant at the higher pressures. At depths greater than 1/4d (d/4) the crown moment is positive, Fig. 6.1. This indicates that the crown response is greatly influenced by the surface boundary at depths shallower than d/4. Overall arching can be applied to the crown at depth, but not at very shallow burial.

Test A-5 (Z = 3/16 in.), Fig. 5.2, agrees very well with test A-1 (Z = 0 in.), Fig. 5.1, both qualitatively and quantitatively, with two exceptions. First, the overpressure required to cause failure is higher for A-5. Second, the invert moment is negative in A-5 and positive in A-1. Again, for the elliptical geometry one would expect this moment always to be positive. However, it appears to be positive or negative in a random variation. This could be a result of geometric imperfections, incipient high buckling modes, or the character and nonuniformity of the soil bedding. The latter, noruniformity of the soil bedding, appears to be the most reasonable explanation at pressure levels below 300 psi. In many tests, A-5 (Z = 3/16 in.), A-2 (Z = 7/16 in.), etc., the moment at the invert changed from negative to positive at pressures gre .er than 300 psi. The significance of the initial bedding decreases as the pressure level increases. An exception is test A-4 (Z = 1-3/4 in.).

Also in test A-5 (Z = 3/16 in.) a vertical diameter increase was recorded at 50 and 100 psi. This is compatible with both the crown and

45

invert moments being negative at that pressure.

Donnellan (1964, p 29) recorded an outward displacement of the radius at the invert of one of his shallow-buried cylinders. The present study recorded only diameter changes, and it is not possible to tell if half a diameter (the radius) increased while the other half decreased.

Test A-2 (Z = 7/16 in.), Fig. 5.3, follows the trends observed at the shallower depths except that no failure was experienced at the maximum machine loading capability of 500 psi. Additionally, the large positive bending moment at the crown observed just prior to failure in tests A-1 (Z = 0 in.) and A-5 (Z = 3/16 in.) was not encountered in this test. Also, the rate of change of moment with pressure decreased, indicating local arching.

Again at about 200 psi the rate of vertical diameter change begins to appear more rapid than below 200 psi. This is probably a result of the cylinder material reaching its yield value at several locations. The moments continue to decrease at overpressures above 200 psi.

Test A-3A (Z = 7/8 in.), Fig. 5.4, exhibits virtually identical thrust values at all four cross sections at pressure below 150 psi. However, at higher levels it establishes the generally observed trend of the spring line having the highest thrust, followed by the crown, with the invert experiencing the least amount of thrust. This is probably a consequence of the bedding providing a soil environment different from that around the crown.

The test (A-3A) was aborted at 400 psi by a broken gas line. The pressure went to zero, the line was repaired, the gages were rezeroed on the oscillograph, and a second test, A-3B, was run without touching the cylinder or the soil. From Fig. 5.5 it can be seen that some aspects of the structural response changed as much as 100 percent as a result of this

cycling of the load. This gives a graphic illustration of how initial, geometric deformations (plastic set in this case) can affect moments. The crown moment is much larger on the second cycle, and the invert moment has changed character greatly.

Test A-4 (Z = 1-3/4 in.), Fig. 5.6, underwent similar response to that of test A-3A (Z = 7/8 in.) with the exception of the invert moment which continued to remain large throughout the test.

The only variable changed between the static tests, A-1 through A-5, and the rapid tests ( msec rise time to 500 psi), A-6 through A-10, Figs. 5.7 through 5.11, was the rise time. The rapid tests in general verified the static tests, but several differences can be seen. First, the pressure necessary to cause collapse was somewhat higher in the rapid tests. This may have been due to a true increase in capacity or to the possibility that some creep mechanism was involved which resulted in failure appearing at a slightly higher pressure in the rapid tests. Second, the values of the thrust are about 20 percent higher in the rapid tests. This may have been due to inertial effects in the soil adding load to the structure. Third, the crown moment is initially positive up to about 100 psi in all rapid tests. For very shallow burial, the moment changed sign and was negative to about 250 psi; then it became positive again. Apparently, the pressure wave struck and depressed the crown, causing the initial positive moment. This occurred at about 3 msec which was slightly greater than the natural period of vibration in the first flexural mode, equation 4.6. This, of course, is much later than would be expected if equation 4.6 were directly applicable.

Although the symmetry around the vertical axis was good, test A-9

(Z = 3/16 in.), Fig. 5.8, illustrates how the spring-line moments can differ by as much as 100 percent (at 150 psi) while the spring-line thrusts agree well. Also, it can be seen that the disparity is not constant during the whole loading cycle, but rather tends to decrease as the cylinder material yields. Also, the moment changes produce deformations which tend to reduce disparities. Test A-7 (Z = 7/8 in.), Fig. 5.10, is a good illustration of the general response.

It is of interest to plot various responses of the group together, as shown in Fig. 6.1. The average spring-line thrust was calculated (refer to Tables 5.2 and 5.3) and the results of all ten tests plotted. It can be seen that all of the test results fall close together and exhibit a linear increase with respect to pressure, and that the rapid test results lie slightly higher (for a given pressure level) than the static results. Data from those cylinders which failed fall right along with those from cylinders which did not fail, indicating that thrust by itself (without some link with depth of burial) will not be an adequate failure criterion for very shallow depths of burial.

The crown moment plot shows how closely the rapid and static tests agree at pressures above 100 psi. The crown moments are always positive at depths greater than d/4.

The average of the crown and invert thrusts was divided by the average spring-line thrust to form the ratio q. This is plotted in Fig. 6.1. After experiencing a large range in values at pressures be'ow 200 rsi, the ratio settles into a band between 0.6 and 0.8. The values are least accurate in the lower pressure regions and are most influenced by the initial conditions created by the soil placement. Disregarding the few very high values,

the trend is to start at about 0.4 (which is approximately equal to the coefficient of earth pressure at rest), increase to about 1.0 as the cylinder began to deform, and then decrease slightly and become relatively constant.

The vertical diameter changes in the static tests are also plotted together. There is a decrease in diameter change with depth of burial for a given overpressure that is noticeable at pressure levels above 250 psi. This reflects the stiffening effect of the soil as the depth of burial increases.

6.1.2 <u>B Group (Sangamon Sand)</u>

The B group differs from the A group only in the value of the yield stresses. The B group had about twice the yield value of the A group.

The pressure causing failure was consistently higher in the B group, Table 5.1, indicating that the yield stress probably had some influence on the collapse pressure. However, this influence does not appear to be large in these tests.

In tests B-1A and B-1B (Z = 0 in.), Figs. 5.12 and 5.13, the effect of cycling is again seen in the character and magnitude of the crown moment. It is also significant that the effect of the cycling is not very pronounced at other locations (which did not yield during first loading). Other studies, Dorris and Albritton (1965) and Albritton and others (1965), have also shown that cycling may not affect the reproducibility more than about 20 percent as long as the cylinder material remains elastic.

Test B-3 (Z = 1-3/4 in.), Fig. 5.16, and test B-4 (Z = 2-5/8 in.), Fig. 5.17, again show that the results are reproducible. They also indicate that moment increases at a decreasing rate (but remains large until the material begins to yield).
The rapid tests, Figs. 5.18 through 5.22, yielded much the same information as the static tests. Tests B-9 (Z = 1-3/4 in.) and B-10 (Z = 2-5/8 in ), Figs. 5.21 and 5.22, illustrate the smoothing out of response that can be expected with deeper depths of burial.

A summary of the B group response is plotted in Fig. 6.2. As with the A group, the spring-line thrust is generally linear with pressure up to a level equivalent to first yielding of the material. The values of rapid test thrusts are larger than those for the static case. The vertical diameter changes fall into a pattern with each other and are lower than those of the A group, Fig. 6.1, at pressures greater than 200 psi. The q values settle into a band between 0.5 and 0.8 for pressures greater than 300 psi.

## 6.1.3 C Group (Sangamon Sand)

The C group of cylinders was only one-twentieth (1/20) as stiff as the A and B groups. The yield stress was high enough that all of the cylinder strains recorded were below the level corresponding to 0.2 percent permanent strain. The pressures required to induce failure were lower than in the A and B groups by a factor of 2 or 3. But, again, at depths greater than one-eighth the diameter no failures occurred. The moments in the C group were substantially smaller, and the moment scale for plotting was changed by an order of magnitude from that used for the B group.

Test C-l (Z = 0 in.), Fig. 5.23, experienced negative moments at all four cross sections and the vertical diameter increased at pressures above 25 psi. This was probably caused by the propensity for collapse in a high-order buckling mode.

The variability in moment response is even more evident in these

very flexible cylinders at shallow burial. Tests C-4 (Z = 3/16 in Fig. 5.24, and C-2 (Z = 7/16 in.), Fig. 5.26, both experienced positive moments at the spring line and the horizontal diameter decreased in C-4. Donnellan (1964, p 26) also recorded inward movement at the spring line of some flexible cylinders. This may be another manifestation of a tendency toward a high-order buckling mode.

The crown thrust was larger than that at the spring line in most of the C group tests. But, q was still less than 1.0 in most cases, Fig. 6.3. The invert thrust was low and probably reflects a decrease in vertical pressure between the crown and invert. This also shows up in a lower arching ratio, Section 6.4.

Rapid tests C-6 through C-9, Figs. 5.28 through 5.31, exhibited the same type curvature changes at shallow burial as the A and B groups. The initial peak positive moment occurred at about 3.5 msec which is about half the natural flexural period given by equation 4.7. Test C-10 (Z = 7/8 in.), Fig. 5.32, is a good example to validate the argument for application of the ring compression theory to flexible cylinders which are not affected by the surface boundary.

Test C-9 (Z = 7/16 in.), Fig. 5.31, exhibited the largest applied pressure, 550 psi, enco.ntered during this investigation. This was the only test in which the maximum pressure deviated from 500 psi. The response ended as usual when the pressure peaked, but the cylinder collapsed about a minute later as the pressure was about to be manually decayed. A stability problem is, of course, very sensitive to slight disturbance, but this also points to a possible creep effect reducing the resistance to buckling. The average spring-line thrust values, Fig. 6.3, show more

scatter than the previous two groups, but the exclusion of test C-2 (Z = 7/16 in.) reduces the spread considerably. Although no characteristics of the test indicated a difference, the results are not in line with the rest of the C group.

The values for the rapid tests are higher than those for the static. The q values for pressures greater than 300 psi lie in a band between about 0.7 and 1.0 with the exception of test C-2. In this test the q values are higher because the spring-line thrusts were lower than the rest of the C group.

## 6.1.4 E Group (Cook's Bayou Sand)

The cylinders used in the E group were identical with those of the A group except that they were cut from a different tube (same nominal material) and hence had a slightly different yield (Appendix A). The three static tests were run as a verification of the reproducibility of the A group results and for comparison with dynamic tests E-4, E-5, and E-6.

The thrust, moment, and diameter change results of E-3 (Z = 0 in.) are plotted together with companion values from test A-1 in Fig. 5.33. The values for thrust are comparable, but the spring-line thrusts of the E group are higher than those of the A group. The diameter change values also are higher and only the spring-line moments are compatible. E-3 failed at 205 psi, whereas A-1 failed at 270 psi. This is reasonably good agreement for such a buckling failure, but the thrust and diameter change trends suggest that the response was more unfavorable in the E test. Different mands were used in the two tests but they have about the same strength and deformation characteristics (Appendix B). If anything, the Cook's Bayou send (E group) is slightly stiffer than the Sangamon send

(A, B, and C groups). As a result, it is felt that the variation in response is a function of the two different methods of placing the sand around the cylinders. The Sangamon River sand was vibrated and rodded in, whereas the Cook's Bayou sand was sprinkled into place. This illustrates one of the difficulties inherent in comparing results from tests in which different placement techniques were used. Conservative conclusions must be drawn.

Test E-2 (Z = 7/16 in.), Fig. 5.34, exhibits the same trends as E-1, and the similarity of the thrust with A-2 is evident. Also, at pressures above 300 psi the moments show closer agreement. It is interesting to note again how the large moments tend to decrease as the cylinder material yields and loading progresses.

Test E-1 (Z = 7/8 in.), Fig. 5.35, exhibits even better agreement with its A group counterpart. However, the large crown moment at pressures below 250 psi and the greater diameter changes of the E group indicate that sprinkling placement of the sand gave a lower density and less restraint.

The recorded values of peak strain on the intrados and extrados for E-5 (Z = 7/16 in.) and E-4 (Z = 7/8 in.), Fig. 5.36, are compared with the values recorded for the static tests at the same 250-psi level (maximum dynamic pressure available). A large amount of ductility is evident in the dynamic tests. Using the analysis outlined by Newmark and Haltiwanger (1962) for a step pulse input of 250 psi and an equivalent elastoplastic resistance function for the cylinders, a theoretical ductility factor of 7 and a theoretical maximum strain of 5100 µin./in. were calculated. This theoretical strain agrees well with the observed strains which ranged between 5000 and 6000 µin./in.

The moment and thrust values are shown in Fig. 5.37. The peak thrusts are uniform around the cylinders for all three dynamic tests at the 250-psi pressure level used. The thrust values for the static and rapid tests are also very consistent with each other, whereas the moment values are widely scattered at the crown and invert.

6.1.5 D Group (Buckshot Clay)

The D group cylinders were buried in clay, but were identical with those of the A and E groups in material and geometry with the exception of a slight change in yield points (Appendix A) resulting from use of different tubes.

The static tests, Figs. 5.38 through 5.42, indicate higher bending moments and larger diameter changes than occurred in sand. The thrust values follow about the same trend as in sand. Generally, symmetric response was recorded and hence opposite gages acted as a check on each other.

The thrusts recorded in several tests, e.g., D-4 (Z = 1-3/4 in.) and D-5 (Z = 2-5/8 in.), were higher at the 45-degree cross section than at the spring line. The instability may very well be concentrated between this level and the crown.

The moments are a highly nonlinear function of overpressure and tend to decrease as the material yielded at high pressure levels, Fig. 5.41.

Ultimate-strength dynamic testing with the WES type Heaviside input is essentially a "go-no go" process. The true failure pressure can only be bracketed between a known collapse and a known survival. A tight bracket would require many tests and be extremely expensive. At the same time it would not be truly reliable because of the inherent scatter in stability problems.

The experience with sand indicated that the rapid and dynamic failure pressures would be relatively close to the static values. This proved to be the case also in clay, and the static failure pressures served as the basis for estimating required dynamic overpressures. The overpressures obtained were not always close to those requested because of variabilities in the loading apparatus. However, a reasonable bracket was obtained for two representative depths of burial, 7/8 in. and 1-3/4 in.

The results obtained from those cylinders which survived are plotted in Fig. 5.43. Results from those cylinders which failed are also plotted to shed more light on what occurred. However, these data should be considered only as guides. They were obtained from the records at incipient failure. This was extremely hard to define for the dynamic tests in which the cylinders failed.

Some instrumentation difficulties were encountered and the data from half the strain gages, Table 5.11, in test D-10 (Z = 7/8 in.) were lost because an oscillograph malfunctioned. However, the thrust values of D-8 (Z = 7/8 in.) and D-6 (Z = 1-3/4 in.) are relatively uniform. The peak moments are at the crown and are positive in sign. The permanent deformations in D-6 and D-10 can be seen from the end views of Fig. 5.46. The strains far exceeded yield in most cases, both in tension and compression, and resulted in high bending moments.

## 6.2 Diameter Change

The diameter changes were small for all tests. In order to verify the validity of the diameter change gages, the cylinder diameters were measured to the nearest one-thousandth of an inch with outside micrometers, both before and after the test (when possible). These

results are plotted in Fig. 6.4 along with the peak diameter change indicated by the diameter change gages. Reasonable verification is evident.

A vertical Collins gage was included in test E-5, and its peak output substantiates the trends.

Several observations can be made based on Fig. 6.4. The horizontal deflection stiffness,  $P_{so}/\Delta_h$ , appears to be independent of the buckling stiffness,  $\frac{EI}{R^3}$ ; but, it varies a great deal with the soil environment. The Sangamon River and Cook's Bayou sands differ by a factor of 2 for horisontal stiffness. The clay is less stiff by an order of magnitude.

Using these empirical values for horizontal stiffness, it is possible to calculate subgrade moduli from the Iowa Formula,

$$\Delta_{h} = \frac{0.166 \, p_{R}^{h}}{EI + 0.061 k_{R}^{h}}$$

vbere

 $\Delta_h$  = horisontal diameter increase, in.  $P_a$  = vertical pressure on top of the cylinder, psi R = cylinder radius, in.

E = modulus of elasticity of the cylinder, pei

I = moment of inertia of the cylinder cross section, in.

 $k_{a}$  = modulus of passive resistance of the soil, 1b/in.<sup>3</sup>

This can be solved for  $k_R$ , E', in terms of the other parameters where E' is called the modulus of soil reaction.

$$E' = \frac{1}{0.061} \left[ (0.166 \text{ R}) \frac{P_a}{\Delta_h} - \frac{EI}{R^3} \right]$$

Substituting R = 1.75 yields

6.1

$$E' = \frac{1}{0.061} \left( 0.2905 \frac{p_a}{\Delta_h} - \frac{EI}{R^3} \right)$$
 6.2

Using the average values of  $\frac{P_{so}}{\Delta_h}$  calculated from the results plotted in Fig. 6.4 as  $\frac{P_a}{\Delta_h}$ , values of E' can be calculated. This assumes no change with depth of burial and is essentially true within the scatter for the range of shallow depths investigated. A trend of increasing stiffness with depth is true of the vertical stiffness. A typical calculation follows.

E' for the A group = 
$$\frac{1}{0.061} \left[ 0.2905(32,900) - 45 \right]$$
  
=  $\frac{1}{0.061} \left( 9550 - 45 \right) = \frac{9505}{0.061} = 155,900 \text{ psi}$  6.3

 $k_{g}$  for the A group =  $\frac{E'}{R} = \frac{155,900}{1.75} = 89,100 \text{ lb/in.}^{3}$  6.4

Also, from equation 6.2 one can compute

These calculations verify how little influence the buckling stiffness of the cylinders has on the deformations in competent soils such as these, under the assumptions of this mathematical model. The deformations are controlled by the stiffness of the soil. For example, in the

57

computations for equation 6.3 the cylinder buckling stiffness,  $\frac{EI}{2}$ , is a negligible term relative to the horizontal soil stiffness  $\frac{P_a}{\Lambda_a}$ .

The calculated soil parameter,  $k_s R$ , is of the same order of magnitude as the moduli from the one-dimensional consolidation and triaxial bests at roughly the same pressures (Appendixes B and C).

Up to this point everything has been analyzed in terms of the overpressure,  $P_{so}$ , on the surface. Here it was assumed that the pressure,  $p_a$ , at the level of the cylinder crown was equal to the surface pressure. This is true by definition only when the cylinder is at zero depth of burial. However, the assumption is satisfactory within the limits discussed in Section 6.3.

# 6.3 Arching Ratio

Overall arching may be assessed by summing forces in the vertical direction above the cylinder. The thrust at the spring line represents a vertical force as does the surface pressure integrated over the area. The arching ratio, AR , is defined as the average spring-line thrust divided by the overpressure times the radius.

$$AR = \frac{\frac{N_{y}(avg)}{P_{gR}}}{P_{gR}}$$
6.5

These ratios have been calculated from the results of the static tests and are plotted in Fig. 6.5.

The A and B groups verified one another well below 200 psi. At that pressure level the A group cylinders began to yield, the moments began to decrease, and hence the cylinders stiffened as a result of approaching more closely a compression mode. The arching ratio increased until such time as the whole cross section yielded, at about 300 psi. After that, the arching ratio decreased.

It appears that the B group began to stiffen at 450 to 500 psi. The moments decreased and the arching ratio began to increase. If the trend were to continue at higher pressure, it would be compatible with the A group behavior.

The E group began with a higher arching ratio than the A group, but at pressures above 250 psi they are similar. These groups had the same buckling stiffness,  $\frac{\text{EI}}{R^3} = 45$ , but as has been pointed out the soil placement techniques differed. This indicates that initial soil differences (densities in the immediate vicinity of the cylinder, Appendix B) created by placement techniques may not be important after the soilstructure system has readjusted under 200-psi overpressure.

It is the writer's opinion that it is appropriate to express cylinder response in terms of the pressure,  $p_a$ , on a horizontal plane through the crown. As a consequence, a correction to  $P_{30}$  would be applicable only if the arching ratio at a given depth varied significantly from the erching ratio at zero depth. This does not occur for the cylinders tested as Fig. 6.5 indicates (although this indication is not conclusive because of the scatter in data for these shallow burials). Hence  $p_a$  and  $P_{so}$  were considered interchangeable.

This does not negate the facts that the arching ratios do differ from group to group at zero depth of burial, and that the arching ratio at zero depth is not necessarily 1.0. For any study of the arching ratio for real structures at depth, it would be necessary first to study the response of the structure at zero depth where a known loading exists. Apparently,

load can be dissipated between the level of the crown and the level of the spring line.

6.4 Ultimate Strength

The collapse pressure, P sof , is plotted in Fig. 6.6 with respect to the stiffness parameter  $\frac{EI}{R^3}$ . The tests of the present investigation, Table 5.1, cover only a small part of the practical range of stiffness and pressure. In order to make the picture as complete as possible, results of other investigations in dense, dry sand are also indicated. The depth of burial is listed next to the symbol in terms of the cylinder diameter, d .

A dotted line indicates the yield value of a high-strength steel in hoop compression for a smooth cylinder. This establishes the upper bound limit of applicability of the elastic buckling theory and hence defines the area of concern for elastic buckling. Above this line the membrane response is inelastic and would be treated in terms of a ductility factor rather than stiffness.

In Figs. 6.7 through 6.10, the collapse pressure has been formed into a nondimensional parameter,  $P_{sof}R^3/EI$ . The test results are plotted in this form with respect to  $\frac{EI}{D^3}$ . A different set of theoretical equations is shown in each of Figs. 6.7 through 6.10. It was mentioned in Chapter 3 that the theoretical equations all contain the cylinder stiffness parameter,  $\frac{EI}{23}$ , as an independent variable.

Open symbols in Figs. 6.6 through 6.10 refer to tests which did not result in failure. Although these tests do not indicate the pressure at which the cylinder would have failed, they are pertinent because they do document areas where failure did not occur.

The amount of data available with which to correlate the clay

results is very slight. Luscher and Hoeg (1964, p 231) reported a series,  $\frac{\text{EI}}{\text{R}^3} = 0.011$ , that experienced failure very similar to their sand tests which were two orders of magnitude higher than the theoretical pressure predicted by the hydrostatic equation,  $p_0 = 3 \frac{\text{EI}}{\text{R}^3}$ , Fig. 6.7. The results of the present investigation,  $\frac{\text{EI}}{\text{R}^3} = 45$ , indicate that the failure pressure for cylinders in clay increases very slowly with increasing depth of burial, Fig. 5.48. The hydrostatic equation is in reasonable agreement with these results, Fig. 6.7, and the results of a test on a stiffer cylinder,  $\frac{\text{EI}}{\text{R}^3} = 82$ , conducted by Dorris and Albritton (1965). On the basis of this, it appears that the hydrostatic buckling equation should be retained for claylike soil media until such time as more experimental evidence fills in the gap between the available data points.

Although far from complete, the data available from tests in dense, dry sand are more plentiful. The present investigation in dense sand showed considerable increase in failure pressure with increase in depth of burial down to d/8, Fig. 5.48. Below this depth failure could not be precipitated with the pressure available, 500 psi. Donnellan (1964, p 42) experienced failures a. d/8 but none at d/4 at 160 psi. However, the conclusion that below some critical depth in dense sand, elastic buckling will not occur is precluded by the results of Bulson (1962) and Luscher and Höeg (1964). But, this conclusion may very well apply to cylinders which are stiffer than some critical stiffness.

The theoretical analysis developed by Luscher and Hoeg (1964, p 143), equation 3.16, is plotted in Fig. 6.8 for several depths of burial. It takes into account the change in soil stiffness with depth and pressure, and predicts the possibility of elastic buckling at depths greater than

d/4 for very flexible structures. The equation fits the author's experimental data and that reported by Bulson (1962) fairly well. For depths greater than d/4, the equation indicates that buckling will not occur before yield of the material for the cylinders used in the present investigation. Hence, this appears to be potentially an adequate design equation for interpolation.

However, for extrapolation of the data a much more conservative approach is in order. A lower bound for these data at zero depth of burial is established by equation 3.8, Fig. 6.9. Substituting  $k_z R = 400$ in equation 3.8,

$$p_c = 40\sqrt{\frac{EI}{R^3}}$$
, psi 6.6

where E is in units of psi, I is in units of in.<sup>3</sup> and R is in units of in. Although the theoretical equation has the hydrostatic buckling value,  $3\frac{EI}{R^3}$ , as a lower bound, it is not possible to say that this would be true for the actual conditions. For a stiff cylinder at very shallow burial, the soil could be a less desirable environment than water because of the nomuniform loading occurring through the soil.

Equation 3.8 with  $k_{z}^{R} = 1400$  fits the writer's data at d/8, Fig. 6.9, and is a lower bound to the data available for more flexible cylinders. Hence, it appears that

$$P_c = 75 \sqrt{\frac{BI}{R^3}}$$
, psi

6.7

would provide a more realistic lower bound to the buckling value than the hydrostatic equation used alone. The units are the same as in equation 6.6.

It is evident that the foregoing values of  $k_z$  are much smaller than those calculated for sand from the lows Formula in Section 6.2.

Equation 3.8 with  $k_{z}R = 37,000$  fits the d/8 no-failure data of the flexible cylinders, and is also shown in Fig. 6.9. It is possible that this may be an appropriate equation for the high overpressure region. This value of  $k_{z}R$  is still lower than those calculated from equation 6.2. If  $k_{z}R =$ 37,000 or higher, it is apparent that buckling will not occur before yield for many practical values of cylinder stiffness (greater than about 1.7) when the cylinder is buried at a depth below d/8. Hence, the theoretical variation of the dense sand properties with respect to pressure may be important only for design pressures below about 500 psi.

The theoretical equation, 3.17, which utilizes Poisson's ratio and Young's modulus of the soil is plotted in Fig. 6.10 for comparison. It follows the general trend of the available test data, but no definite conclusions can be drawn.

A comparison of the results of the A and B groups, Table 5.1, indicates that the cylinder strength may play a part in the buckling values. This is probably a reflection of the decrease in effective buckling stiffness which occurs when part of the cross section yields. However, the failure values between the groups did not differ by more than 25 percent although the yield values varied by a factor of 2.

The catastrophic manner in which the cylinders failed is probably a consequence of the large amount of strain energy in the cylinders at incipient collapse. Figs. 5.44 and 5.45 depict the failed cylinders. The irregularities in the postbuckling shapes were caused by the cylinder crowns striking the longitudinal rods (which connected the end caps) as they caved in. The postcollapse configuration in city is shown in Fig. 5.47.

CHAPTER 7. SUMMARY, CONCLUSIONS, AND RECOMMENDATIONS

7.1 Summery

Forty-six, small, horizontally oriented cylinders were tested in two kinds of soil media: dense, dry sand and stiff clay. The applied overpressure, vertical, and horizontal diameter changes for the static tests and hoop strains were measured. The cylinders were all made of aluminum. Three alloys were involved having yield stress values of 7,500, 12,700, and 42,100 psi. The cylinders had identical outside diameters of 3.5 in. and two thicknesses, 0.022 and 0.065 in. Hence, the cylinder stiffnesses,  $\frac{\text{KI}}{r_3}$ , were 1.7 and 45 (d/h = 159 and 54), respectively.

The test structures were buried at depths ranging from zero to three-quarters of the outside diameter, 2-5/8 in. Three overpressure rise times were used: a static rise time (10 to 15 min), a rapid rise time (13 msec), and a dynamic rise time (0.3 msec).

The relations between stress, thrust, moment, and diameter change were plotted and analyzed with respect to the surface overpressure. The pressure necessary to cause collapse was established and compared with several theoretical solutions and with the results of other investigations. The horisontal and vertical stiffnesses as indicated by the diameter changes were analyzed and compared with theoretical concepts.

It was not possible to collapse cylinders of either stiffness when buried in sand at depths equal to or greater than one-eighth the diameter, 7/16 in., under the available 500-psi pressure. In stiff clay, however, it was possible to define collapse even at the deepest burial, three-quarters of the diameter or 2-5/8 in.

## 7.2 Conclusions

All of the conclusions are based on the assumption of the planewave loading which was used during this investigation.

# 7.2.1 Cylinders in Dense, Dry Sand

The difference between static and rapid loading in the elastic response of the cylinder is small (within 20 percent). The rapid loading was observed, Figs. 6.1 through 6.3, to cause larger thrusts.

Inelastic strains are much higher under dynamic loading than under static or rapid loading at the same pressure, Fig. 5.36. However, a cylinder buried at a depth greater than one-eighth its diameter can sustain large inelastic bending strains without experiencing structural failure or collapse.

Based on an equivalent elastoplastic resistance function for the cylinder and an approximate step-pulse loading, a ductility factor of about 7 was found to be conservative for the dynamic tests. No failures occurred, so it is not possible to say what the ductility factor to define failure would be.

Thrust is generally a linear function of surface overpressure. It is largest at the spring line, smaller at the crown, and smallest at the invert. For overpressures greater than 200 psi, the average value of the horizontal force divided by the vertical force on the cylinder is shout 0.8. However, the hoop compression theory appears to be adequate for design.

Noment is generally a nonlinear function of surface overpressure. It tends to increase at a decreasing rate (probably governed by local arching from point to point around the circumference of the cylinder), until the cylinder material begins to yield. Thereafter, the moments tend to decrease. The moments are larger in the stiffer cylinders. A depth of burial of one-eighth the diameter is a critical depth for the sign of the crown moment. At shallower depths the curvature increases, whereas for deeper depths the moment is positive and the curvature tends to decrease.

For zero depth of burial, the pressure to cause buckling failure can be defined by

$$p_{cr} = 40 \sqrt{\frac{EI}{R^3}}$$
, psi 7.1

where E is in units of psi, I is in units of in.<sup>3</sup>, and R is in units of in. This is an empirical fit of equation 3.8 to the test data with  $k_z R = 400$ , Fig. 6.9. For depths of burial equal to or greater than oneeighth the diameter, the pressure to cause buckling failure can be bounded until more experimental data becomes available by

$$\mathbf{p}_{cr} = 75 \sqrt{\frac{BI}{R^3}}$$
 7.2

where the units are the same as those in equation 7.1. This is equation 3.8 with  $k_R = 1400$ . Failure occurs (at the shallow burial) by a sudden snap-through of the crown. The result is a complete collapse. But, no collapse could be induced at depths greater than one-eighth the diameter for  $\frac{EI}{-3} \ge 1.7$  for pressures up to 500 psi.

Depths of burial greater than one-eighth the dismeter probably have more significant effects (on elastic buckling) than indicated by the allowable pressures from equation 7.2. However, since the effects of the depths were not satisfactorily defined because no failures occurred, they can only be considered as an additional factor of safety. Equation 7.2 represents points where no failure occurred and does not define failure.

However, this is a more realistic equation than the hydrostatic prediction. It is hypothesized that equation 7.2 is still overly conservative for values of  $\frac{EI}{D3}$  greater than about 1.7.

It is not possible at present to identify adequately the appropriate soil properties controlling cylinder collapse with soil properties obtained from standard laboratory tests.

The technique used to place the sand in the vicinity of the cylinder can affect the response of the cylinder and apparent deformation stiffness by as much as 50 percent. However, the pressure required to cause collapse differs by only  $\pm 25$  percent. Sprinkling in the vicinity of the cylinder is less effective than vibrating or rodding.

The arching ratio (defined as the average spring-line thrust divided by the overpressure times the radius) for cylinders buried with the crown tangent to the soil surface is not necessarily 1.0.

7.2.2 Cylinders in Stiff Clay

Collapse of the cylinder occurs by a sudden snap-through of the crown. Regardless of the depth of burial, this mode of failure occurs even at the maximum depth tested, three-quarters of the diameter.

Only a small increase in failure pressure results from an increase in depth of burial. The hydrostatic buckling equation

$$P_{cr} = 3 \frac{KI}{R^3}$$

was appropriate for the cylinders used, Fig. 6.7, and should be slightly conservative for cylinders buried at depths greater than one-eighth the diameter. This equation implies a low value of  $k_{\rm s}R$ .

Noments and deformations of the cylinder were much larger than

67

7.3

in sand at comparable pressures. They were both highly nonlinear functions of pressure.

7.3 Recommendations for Future Study

High pressure tests (500 psi or greater) should be conducted in dense sand with cylinder stiffnesses,  $\frac{EI}{R^3}$ , between 0.1 and 1.0 for the purpose of establishing failure pressures for depths of burial greater than one-eighth the cylinder diameter. Materials with high yield strengths, such as high-strength steel or aluminum, would best serve the purpose. Elastic buckling could be isolated relative to the buckling stiffness without consideration of the reduced stiffness due to yielding.

Some ultimate strength tests should be conducted with relatively large (2-ft-diameter) cylinders in the WES Large Blast Load Generator to investigate the possibility of size effects. These should have the same value of  $\frac{\text{EI}}{\text{R}^3}$  as some smaller diameter cylinders discussed in the literature, or else small companion cylinders should be tested concurrently.

A cylinder with  $\frac{51}{R^3} = 220$  should be tested at zero depth of burial in dense sand at pressure greater than 500 psi to extend the range of knowledge of equation 7.1.

The work on elastic buckling should be done with static loading (but fast enough that longtime effects such as creep do not enter) to gain the most for the least cost. Selective dynamic testing should then be done to assure the applicability of the knowledge gained from the static tests.

Once the limits of the buckling problem are established, then dynamic studies should be conducted to determine an appropriate magnitude for the ductility factor to define collapse in the nonelastic region of

cylinder response. Since yielding is not a proper criterion for failure, it is doubtful that the studies of the elastic response of cylinders will shed much light on the ultimate strength except when buckling governs.

CARLES CARLES & FRANKLE

Once the dense, dry sand-cylinder interaction is fully understood, other soil environments such as medium density (relative density of 50 percent), and partially saturated sands should be investigated. It may then be possible to develop a single equation which can take into account the significant soil properties in a realistic manner.

Concurrent with the foregoing, an attempt should be made to determine the pressure distribution on the surface of the buried cylinder from the measured strains. The solution by Riley (1965) for WES which expresses the load in a Fourier series with undetermined coefficients could be used.

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Table 4.1

Geometric and Material Properties of Test Cylinders  $\mathbf{E} = 10 \times 10^6$  psi; t = 10.5 in.; d = 3.5 in.; t/R = 6

(Pail)       (Im.)       (Im.) <t< th=""><th></th><th>°y1</th><th>22</th><th>а</th><th>×</th><th>r Ha</th><th>I</th><th>ᆀᆢ</th><th>Ð</th></t<>		°y1	22	а	×	r Ha	I	ᆀᆢ	Ð
5000         7600         0.065         1.72         2289         228.9         45.0         54           3h00         6100         0.065         1.72         2289         228.9         45.0         54           4500         7h00         0.065         1.72         2289         228.9         45.0         54           11,000         12,700         0.065         1.72         2289         228.9         45.0         54           37,500         kz,100         0.062         1.74         89         8.9         1.77         139		(fai)	(Fsg)	(iii)	( <del>.</del> )	( <u>c.nt °01)</u>	( <u>चा-वा)</u>	( <u>F84</u> )	14
34:00       61:00       0.065       1.72       2289       228.9       45.0       54         45:00       74:00       0.065       1.72       2289       228.9       45.0       54         11,000       12,700       0.065       1.72       2289       228.9       45.0       54         37,500       42,100       0.022       1.74       89       8.9       1.7       159		805	0092	0.065	1.72	2289	228.9	45.0	Ŧ
4500         7400         0.065         1.72         2289         228.9         45.0         54           11,000         12,700         0.065         1.72         2289         228.9         45.0         54           37,500         42,100         0.022         1.74         89         8.9         1.7         159	· · · ·	3100	6100	0.065	1.72	2289	228.9	45.0	ま
<b>11,000</b> 12,700 0.065 1.72 2289 228.9 45.0 54 <b>37,500 42,100</b> 0.022 1.74 89 8.9 1.7 159		<b>#</b> 500	2400	0.065	1.72	2289	228.9	45.0	5
<b>37,500 k2,100 0.022 1.74 89 8.9 1.7 159</b>		ш,000	12,700	0.065	1.72	2289	228.9	45.0	54
		37,500	42,100	0.022	1.74	&	8.9	1.7	159

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Course South Londing Londing Cyt- Band Bratic A-1 Band Bratic A-1 Band Bratic A-1 Band Bratic A-1 Band Bratic A-1 Band Bratic A-10 Band Bratic C-1 Clay Static C-1 Clay Static P-1 Clay Branch	2 3 2 3 6 5 3 9 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	171- 1114er A-5 A-9						Z = 7	8 in.		2/2 1n	- 2-1	5/X 1 m
A Band Btatic Band Btatic Band Btatic Band Btatic Band Btatic Band Btatic Band Btatic Band Static C-6 Clay Static C-6 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Static P-10 Clay Prantic Prantic P-10 Clay Prantic Prantic P-10 Clay Prantic Prantic P-10 Clay Prantic Prantic Prantic P-10 Prantic Prantic Prantic Prantic P-10 Prantic Pr	5 1 3 1 3 3 3 3 3 5 5 5 5 5 5 5 5 5 5 5	A-5	50 18 1	Cy1- Inder	0.9 4	Cy1- ther	P so ps1	Cyl- Inder		Cyl -	P 80	Cy1-	P so
A Sand Rupid Bank Rupid Bank Rupid Bank Rupid Bank Rupid Bank Rupid Bank C-6 Clay Static Clay Branic Clay Branic C-6 P-1 Clay Branic C-6	36 13 8 19 36 37 36 37 36 37 36 37 36 37 37 36 37 37 37 37 37 37 37 37 37 37 37 37 37	6-V	325			A-2	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	A-3	1005	A-4	¥00;5		
Beend Static Beend Static Beend Static Beend Static Beend Static Clay Static Clay Static Clay Dynamic Clay Dynamic Dyn	N 3 8 E 2 %		84			A-8	500##	A-7	200**	<b>A-</b> 6	200**		
Clay bynamic R-6 Clay bynamic R-6 Clay bynamic R-6 Clay bynamic R-6	8 E 2 2 4					Ъ-5	\$1005	<b>B-</b> 2	500#	B-3	1005	₽-4 ₽	£005
C Baund Static Baund Static Baund Static Baund Dynamic R-6 Clay Static Clay Bynamic R-6 Clay Dynamic P-1	8 E 8 F					<b>B-</b> 7	500##	<b>B-8</b>	500##	<del>Ч</del>	¥#005	<b>B-</b> 10	*005
C Search Reptid Band Static E-6 Band Dynamic E-6 Clay Static P-1 Clay Dynamic P-1	55 F	7-5	195	5-5	<del>200##</del>	2-10	500##	C-3	£005				i.
<b>Band Static E-3</b> <b>Band Dynamic E-6</b> <b>Clay Static D-1</b> <b>Clay Dynamic D-1</b>	205 254**	6-1	350	с-8	200	c-9	200	C-10	¥*005				
B Sand Dynamic 2-6 Clay Static D-1 Clay Dynamic	5274	• • • • • • •				E-2	1004	E-1	140**				
D Clay Static D-1 Clay Dynamic						E-5	262**	E-4	264**				
Dynamic	8	•				D-2	130	D-3	100	D-4	190	D-5	180
								D-10	**16	<b>D-</b> 6	160**		
								<b>D-</b> 8	**9TT	D-7	180		
								<b>6-</b> 0	148				
	n												
		•											

or and the

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			Table 5.2						
www.	Nonest .	rad	Deflection;	Juata	A-:.	A-2,	A-3A,	A-38, A-	•. +5
		-	Om	erpres	sure,	381			
	50	20	250		<u> </u>		<u>100</u>	<u> </u>	

Cinge	Neesurvalent	<u>- xo</u>	100	150	210		<u> </u>	<u></u>
					Test A-1	(2 - 0 10	<u></u> )	
1	Strain, win./in.	194	396	485	¥85	276	-523	
	Stress, pai	1910	3960	4850	4850	2760	-5102	
18	Stress, psi	200	232	1450	4080	7143	••	
1-1 <b>a</b>	Thrust, 1b/in.	70	146	205 J 20	290	3		
		-0.01		Th:	141	142	121	
ذ	Stross, psi	****	660	1410	1610	-320	1230	
34	Strain, win./in.	100	1740	260 2600	327	4540 4540	480 4800	
3-3a	Thrust, 10/in.	47	85	124	152	190	196	
	Moment, in -ib/in.	0.20	0.30	0.35	9.05 Acc	ال الله. الله ال	1.40	
2	Strein, sis./in. Stress. pri	183 1830	319 3190	4750	5864	6807	7125	
26	Strain, uin./in.	-18	14	56	105	550	890 6062	
2-2	Stress, pai Thrust, 10/18.	-160	106	in	268	NOF	434	
	Mument, in16/in.	-0.71	-1.0/	-1.48	-1.79	-0.53	-0.37	
4	Strain, wim./im.	125	224	345 3440	581 5360	6286	6623	
44	Strain, Min./in.	9	51	102	191	404	510	
h-ha	Stress, pei Though lb/in	90 Ma	510	1020	1910	363	308	
	Noment, in1b/in.	-0.41	-0.61	-0.95	+1.29	-0.54	+0.59	
DC1	Deflection, is.	0.001	0.001	0.003	0.007	0.017	0.037	
101213-36	Seriection, in. Any thrust, lb/in.	59	116	165	201	291	••	
2-2614-46	Avg thrust, 1h/in.	49	, 99	159	258	363 0.76	411	
	4	1.00	1.11	2. v=	v			
					Test A-5	(2 + 3/1)	18.1	
7	Strain, Hin., .n.	290 2900	953 5216	632 5568	554 5240	230 2380		-5062
la	Strein, Muis./in.	0	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	133	401	1962		6830
1.1.	Stress, pel	0	0	1330	307	7201		9903
	homent, in1b/1z.	-1.02	-1.92	-1.59	-0.15	1.27		3.03
3	Strain, sin./in.	261	296	<b>361</b>	. 256	275		262
<b>L</b>	Stress, pet Streis, six./is.	1410	10 10	atu چ	106	182		279
	Birest, pti	100	100	350	131	1820		2590 169
3+38	Noment, inib/in.	-0.16	-0.80	0.85	0.6	-0. 32		-0.01
2	Strain, win /in.	113	261	356	535	1005		8145
•	Struss, pti	1130	2430	3560	5456	6285		1610
	Street, pti	ē.	310	780	1600	3108		634-2
3-23	Depust, 10/1a.	-0.10	.0.75	-1,00	-1.29	- 317		.0.13
	Strain, sin./in.	162	384	548	111	1290		217
	Stress, pet	1620	<b>SHO</b>	2000	1024	199		7341
-	#17618, 018-718 #17644, 171	130	-110	¢.	\$50	3890		6501
5-46 ···	hundt, ik/in.	1	100		200 St.	366		374 
	Billion, includes		-0.000	5.2003	0.005	5.017		2.0 <b>1</b> 9
94.1 962	Befinting in	0.001	0.001	5.002	0.00	2.00		a. 📬
- iat )- )a	Any thrust, 16/18		100	173		170		139
Set all and	A	. 1.80	4.5	1.12	7. 🗰	0.71		¢. <b>6</b> 8
i di		÷.,			Inc. Ad	12 + 1/1	i lead	
	Areas	194	347	<b>3</b> /7	516	539		
	Breas phi	1984	1970	9031	5)40 ***	20	•	(15) (15)
1	HIPLIA, MAR / LA	<b></b>	· · ·	ista	Nille -	6204		7.75
l+in 👘	Biret, 1615		170	2) h		2. <b>6</b>		* * <b>1</b> 1
	Banks			100		-		M
	Portes, phil	. sales	-	<b></b>	10	-		3003
٠	Maria, sis /is	1. 100	17 170	- Note	And State	2.40		1140
	Perest, Hyla.	57	107	14-3				
e Maria de Cara		407. <b>57</b>	493. <b>93</b>	-1.3XW	-3-34 <b>7</b>			***
t 📕 e se s	FLIGHT HER / SR.		1	-	WH.			83 
<b>#</b>	Strain, and /10	- 10			110	TLI Vicity		1994A
	Brut, Bysa.	5	110	100				
	Bin.st, 1818/18-		-1,4	<b>63.40</b> 2	*1.14	- <b></b>		-
. <b>*</b>	Stable, etc./50			4.77 6 <b>462</b>		ہ3		
1 <b>46</b> - 1	Staula. als /is	-43			116			. <b>194</b>
المعد الم	States, ret	4 <b>3</b> 0	- 2000					44 J :
	Support, 18176/18.		+1.42	-1.93		-4. TV		ی <b>دہ</b> محمد نے
<b>10</b>	buffaction, in	C.444	0.000		9.5350 3.555	9.000 2.000		9.0 <b>11</b> 9.945
je zastie in	Ang Marut, Main	4. <b>U</b>	129		2µ	2		194
-	ang tayont, ily's.	1.14	114 1.13	178	- <b>199</b>	9. YY		

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Scrain, Streas.

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4.23	5.0	4 12	5.97
<b>44</b>			•••
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3-8-	1 <b>4</b> 4	44	••
	·	**	**
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<b>24</b>	**	**	· • •
<b>Set</b>	44	** -	
44	**	ale:	· • • ·
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9. <b>ANT</b>	4.988	3 ( <b>1</b> 1	8 <b>39</b>
3.647	8 349	S. 183	
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Gage	Measurement	50	100	150	200	250 270	<b>e</b> , psi <u>300</u>	325	350	400	450	500
					Test A-	3A (Z = 7/8 in.)						
1	Strain, µin./in.	88	169	213	199	213	389		940	1362		
18.	Strain, Jin./in.	380 _ <b>8</b> 7	174	239	463	820	3890 1 <b>38</b> 0		5934 2761	6745 31.25		
1-14	Stress, psi Thrust, 15/1n.	870 57	1740 113	2890 163	4630 215	5899 300	6759		7365 444	7033 477		
	Moment, in1b/in.	0.00	0. <b>72</b>	0.27	0.93	1.36	0.80		0.47	0.37		
3	Strain, ui/in. Stress, psi	166 1660	250 2600	367 3670	464 4640	560 5267	647 5595		796 5857	954 6135		
3 <b>a</b>	Strain, pin. An.	31	55	80	128	185	222		332	473		
3-34	Thruat, 15/in.	64	209	145	192	240	273		327	367		
2	Moment, in15/in.	-0.48	-0.79	-1.01	-1.18	-1.27	-1.26		-0.86	-0.41		
	Stress, psi	1710	3080	4260	5285	6815	7593		8337			
28	Stress, pai	-14 -14C	25 260	110	262 2620	1170 6515	2341 7409		3745 8138			
2-2 <b>n</b>	Thrust, 1b/in. Moment, in1b/in.	51 - 0.65	109 -0.99	174 -1.11	266 •0+-	435	488 +0.06		505 -0.07			
4	Strain, win./in.	213	368	530	697	1198	2243		3545	4600		
48	Stress, psi Strain, uin./in.	2130 -46	3680 -20	5133	5683	6564 435	7355 1435		8040	8558 36(1)		
h_he	Stress, psi	-460	-200	330	1130	4350	6802		7550	8067		
4-48	Moment, in1b/in.	-0.91	-1.37	-1.74	-1.73	-0.62	-0.20		-0.17	-0.17		
DC1	Deflection, in.	0.005	0.007	0.009	0.013	C.021	0.029		0. <b>039</b>	0.048		
1-1a:3-3a	Avg thrust, 1b/in.	61	110	154	204	270	0:013 320		386	0.015		
2-2a:4-4a	Avg thrust, 1b/in.	53 1.15	1)1 0:99	179 0. <b>86</b>	258 0.79	407 0.66	475 0.69		521 0.74	540 0.79		
					Test A-	3B(L = 7/8 in.)						
1	Strain, uin./in.	-206	-244	-264	-257	-264	-198		_ أؤني	66	294	896
la.	Stress, psi Strain, µin./in.	-2060 338	-2440	-2640 695	-2570 840	-2640 1070	-1980 1331		-440 1689	660 2045	2940 2620	6033 3702
1_1.	Stress, psi	3380	5093	5679	5934	6339	6720		7005	7245	7561	8117
	Moment, in1b/in.	1.92	2.69	3.16	3.27	3.35	3.05		2.32	1.52	1.11	0.64
3	Strain, uin./in.	79 790	122	175	21c	271 271	3060		376	507	655	- <u>814</u>
3 <b>a</b>	Strain, µin./in.	55	117	184	234	296	362		474	621	817	1056
3-3a	Stress, ps1 Thrust, lb/in.	550 44	1170 <b>78</b>	1840 117	2340	2960 184	3620 217		4740 276	5536 344	5894 374	6315 397
_	Moment, in1b/in.	-0.08	-0.02	0.03	0.06	0.09	0.20		0.35	0.18	0.10	0.15
2	Strain, µir./in Stress, īsi	216 2160	368 3680	518 5080	650 5600	800 58 <b>6</b> 4	920 6075		1200 6568	2570 7534	4010 8268	
28	Strain, pin./in.	69 690	131	220	302	392	488		742	1971	3302	4420
2-24	Thrust, b/in.		162	240	298	341	366		401	479	526	
L.	Strain, uin./in.	-0.52 100	زه. <i>ب</i> احدد	-1.04 LAR	-0.95 500	-0.52	-0.57 860		-0.28	-0.12	-0.12 -09.60	
	Streas, psi	1990	3380	4680	5400	5751	5970		6251	7166	7680	6317
48.	Streac, psi	470	100	174	2 <u>34</u> 0 2340	315 3150	388 3880		515 5067	1375 6755	2300 7386	3362 7950
<b>կ₋կ≞</b>	Thrust, lb/in. Moment, inlb/in.	80 -0.54	142 -0.84	209 - 1.04	264 -1.14	315 -0.93	346 =0.65		375	453	490	529
DC1	Deflection, in.	0.007	3.011	0.013	0.016	0.020	0.022		0.026	0.034	0.043	0.053
DC2 1-lat3-3a	Deflection, in. Avg threat, lb/in.	0.004 հե	0.007	0.008	0.009	0.011	0.013		0.014	0.015	0.016	0.017
2-2a:4-4a	Avg thrust, 1b/in.	87	152	225	281	325	355		388	466	508	529
	q	0.91	0,99	0.50	0.56	(7) 3 2 4 1 3	0.67		0.78	0.7?	0.78	0.83
1	Strain, uin./in.	o	30	78	16st A-4	$(2 = 1 - 3/4 \ln n)$	295		507	780	1162	1525
1.	Stress, psi	0	390	780	1430	1950	2950		5031	5829	6466	6883
	Stress, psi	1340	2010	2950	443 4430	551	900 6040		1290 6688	1706 7019	2226 7346	2760 7636
1-1 <b>4</b>	Thrust, 15/in. Noment. in15/in.	եկ 0.47	78 0.57	)21 0.76	190 1.06	261 1,35	330		391 0.53	425	454	474 C. 26
3	Strain, µin./in.	166	280	402	543	744	963		1226	1480	1813	2035
3.	Stress, pai Strain, uin.//n.	1660 - 38	2800	4020	5191	5765	6151		6614 - 104	6839 -86	7104	7241
	Stress, psi	-380	-570	-760	-1140	-1230	-1230		-1040	-860	-380	380
3-38	Moment, in1b/in.	-0,72	-1.19	-1.68	-2.30	-2.68	-2.76		269 -2.67	-2.54	337	-1.93
2	Strain, µin./in.	267	445 http://	609	807	1275	2140		3322	4470	5650	690()
28	Strain, µin./in.	-41	-41	-10	10	0070 134	7298 909		7930 2029	8494 3043	9023 1110	9558 مراجع
2-24	Stress, psi Thrust. 1b/1n.	-410 73	-410 131	-100	100 241	1340 331	6056 ціла		7235	7789	8317	88:14
	Moment, in1b/in.	-1.08	-1.71	-2.08	-2.19	-1.68	-0.41		-0.25	-0.25	-0.25	-0.24
4	Strain, µin./in. Strass. pai	1944 1940	296 2960	445	571 5316	765 5802	1267		2238	3220	4293	54.24
lia.	Strain, µin./in.	-22	-11	34	122	290	793		1767	2688	3707	4725
4-4-6	Thrust, 1b/in.	-220	-110	340 156	223	313	407		470	7597 50 <b>3</b>	8119 537	5620 571
	Noment, in16/in.	-0.76	-1.08	-1.45	-1,52	-1.03	-0.29		-0.09	-0.10	-0.10	-0.11
DC2	Deflection, in.	0,004	0.003	0.005	0.007	0.010	0.016 0.010		0.021 0.011	0.028	0.034	0,044 0,013
1-14:3-34 2-24:4-4+	Avg thrust, 1b/in.	13	75	114	165	225	279		330	363	396	419
	3	0.66	0.67	0.66	0.71	0.70	0.66		0.68	0.70	0.72	0.72

15

Table 5.3 Strain, Stress, Thrust, and Moment; Tests A-6, A-7, A-6, A-9, A-10

	Mananana	50	100	150	0v	erpressure,	100	350	400	450	500
<u>    Gare    </u>	Mexsurement			 	<u></u>	<u></u>			400		
•	Strain, uin./in.	-149	432	894	1281	461	-1.830		л.		-
	Stress, psi Strein uin /in.	-14 40 3077	4320	6029 26	6681 309	4610 3942	-8676 10400				
19	Stress, psi	3790	510	260	3090	8235 465	10950 274	· · ·			
i-1a	Moment, inb/in.	1.61	+1.34	-2.14	-1.02	0.95	8.24				
3	Strain, µin./in.	-80 -800	151 1510	330 3330	4460 4460	423 4230	503 5013				
3a.	Strain, µin./in.	231	284	328	389	458	503 5013		-		
j-38	Thrust, 1b/in.	49	141	214	271	286	326			-	
2	Noment, in1b/in.	1.09 260	0.47	-0.01 808	-0.20 h=7	:F30	2662				
2	Stress, poi	2690	3160	3980	4570	6958	7691				
28.	Stress, psi	-120	1040	1580	3450	6630	7193				
2-2a	Thrust, lb/in. Moment, inlb/in.	-0 <b>-</b> 94	137 -0.75	194 -0.70	262 ~0.38	442 -0.11	-0.18				
4	Strain, µin./in.	275	375	462	713	1338	2564 2504				
a	Stress, psi Strain, µin./in.	2750	3750	4620	353	1014	1276				
Laba	Stress, psi Thrust, lb/in.	-110 86	800 148	1590 202	3530 322	6241 423	6677 464				
	Moment, in1b/in.	-1.01	-1.64	-1.07	-0.77	-0.18	-0.30				
1-3a:3-3a 2-3a.4-43	Avg thrust, 1b/in. Avg thrust, 1b/in.	51 85	149 143	237 195	318 292	376 433	474				
	4	0.60	1.04	1.20	1.09	0. <b>8</b> 7					
				Test /	<u>1-9 (2 = 3/16</u>	<u>5 in.)</u>	- 71.0	1 1 9 5	17778		
1	Strain, µin./in. Stress, psi	-2070	4450	6840	7586	7466	7052	6542	-7076		
la	Strain, µip./in. Stress, psi	<b>3</b> 24 3240	162 1620	-162 -1620	81 810	1462 5825	59 <b>82</b> 9150	10830			• '
1-la	Thrust, 1b/in.	38	197 -1.00	283 -2,86	398 -1,68	466	531	580 1.42			·
3	Ctrain, µin./in.	35	263	518	658	702	676	711	737		
20	Stress, psi	350	R630 98	5080 82	5614 98	5692 155	5646 221	5707	5753 31 i		2
22	Stress, psi	1310	960	820	9 <b>8</b> 0	1550	2210	2620	3110		
3-32	Moment, in1b/in.	0.34	-0.58	-1.53	-1.75	-1.57	-1.29	-1.14	-0.94		
2	Strain, µin./in.	261 2630	2 <b>8</b> 9	361	602 5454	1133	2699 7603	3687 8109	4 <b>8</b> 92 8702		
2a.	Strain, µin./in.	25	0	199	373	1772	2414	3:034	4505		
2-2a	Stress, psi Thr st, lb/in.	250 86	. 0 94	1990	3730	399	489	526	559		•
	Moment, in1b/in.	-0. <b>7</b> 6	-1.02	-0.57	-0.62 850	-0.22 1560	-0.05	-0.01	-0.07 5670		
<i>i</i> .	Strain, µin./in. Stress, psi	330	3560	5262	5933	6814	7984	859 251	9031		
-B.	Strain, µin./in. Stress, noi	-49	0 0	74 740	221 2210	959 6144	2754	3011 8170	8042		
4-48	Thrust, lt/in. Moment in -lh/in.	91 -1-33	116 -1.25	204 -1.67	304	425 -C.24	508 -0.12	540 ≈0.10	575 -0.13		
1-1a:3-3a	Avg thrust, 1b/in.	46	157	239	318	365	405	439	*-		
2-2a:4-4a	Ave thrust, lb/in.	89 0.52	105 1.50	1:3	307 ⊥.04	412 0. <b>89</b>	- 499 9. <b>81</b>	0.82			÷.,
	•			Test	A-B(2 = 7/1)	6 in.)				1.	
i	Strain, µin./in.	-220	422	906	1293	1188	1139	1355	1557	1698	1707
la	Stress, psi Strain, uin/in.	-2200 249	4220 -28	6051 166	690 443	6547 1773	6454 3574	6739 5154	7094	9726	15303
1-14	Stress, pai Thruce th/in	2490	-280 128	1660 299	4430	7072 444	8054 420	8818 511	9618 545	10703 584	11569 618
1-:4	Moment, in1b/in.	1.65	-1.58	-1.55	-0.64	0.17	0.52	0.71	0.93	1.26	1.58
3	Strain, µin./in. Stress. psi	7#6 041	474	849 5950	1236 6631	1529 6:78	1804 7097	2133 7293	2437 7461		
За.	Strain, µin./in.	89	33	17	50	127	283 2830	371	533 5147	749 5774	1004
3-3a	Thrust, 1b/in.	76	165	250	308	349	390	414	435	••	
-,	Moment, inib/in.	-0.20	-1.50 1.90	-2.17 620	-∠.0a 787	2290	4080	-0.93	7348	9329	11333
-	Stress, psi	3810	4290	5534	3841 har	7381	8303 320	8956	9723	10539	11252
28	strain, uin./in. Stress, psi	25 250	1250	1760	4270	6923	7918	8549	5.94	9931	10636
2+28	Thruct, ib/in. Moment, in1b/in.	-1.25	1 <b>8</b> 0 -1.07	253 -1,41	347 -0.49	466 -0.16	-0.14	-0.14	-0.19	-0.21	-0.2)
4	Strain, µin /in.	380	456	608	836	2155	3677	4945	6568	8242	9890
1.4	Stress, psi Strain. win./in.	3800	4560 100	5480 201	5927	7307 1413	2926	4305	5685	7367	8611
- 4-b-	Stress, psi	0	1000 181	201.0 258	4370	6786 459	7726 515	8413 597	9037 599	9731 644	10243
	Moment, in1b/in.	-1.34	-1.25	-1.29	-0.47	-0.19	-0.13	-0.11	-0.13	-0.13	-0.19
1-1 <b>4:</b> 3-34	Avg thrust, 1b/in.	43	147	275 256	347 350	397 463	435	4 <b>63</b> 563	490	655	69
 	o wak cutuze, tolige	0.34	0.81	1.07	0.99	0.86	0.83	0.82	0.81	••	**

(Continued)

## Table 5.3 (Concluded)

rain, µin./in. reas, pei russ, pei rust, lk/in. rust, lk/in. reas, pei reas, pei reas, pei rust, lb/in. reas, pei rust, lb/in. reas, pei rein, µin./in. reas, pei rust, lk/in. reas, pei rust, lk/in.	-155 -1550 2510 2510 311 1.43 1.85 1.850 46 0.61 258 2580 35 2580	103 1030 275 2750 123 0.61 61 61 61 610 245 2450 99 0.65	<b>Test</b> 415 415 450 461 461 461 285 0.16 217 2170 2170 161 0.21 3	A-7 (2 = 7/4 675 5644 769 5809 372 0.06 3300 3300 3300 3305 3050 206	3 1n.)           766           5804           1271           6673           406           0.31           391           3920           3760	831 5919 1547 422 0.35 400 425	1078 6353 2268 7369 453 0.33 478 4780 528	1389 6767 2972 7751 475 0.34 530 5133 659	1857 7139 3839 81.84 499 0.37 626 5558	2338 7407 4365 8443 516 0.36 .78 5649
rain, µin./in. reas, pei russ, pei russ, Jé/in. russ, Jé/in. reas, pei rain, µin./in. reas, pei rust, lb/in. rust, lb/in. reas, pei rein, µin./in. reas, pei reas, pei russ, pei	-155 -1550 2510 31 1.43 .440 1850 46 0.81 258 2580 35 350	103 275 2750 2250 245 245 245 245 245 245 245 245 245 245	415 415 461 461 285 0.16 217 2170 2170 2177 21770 161 	675 5644 769 5809 372 0.06 3300 3300 3300 3305 3050 206	766 5804 1271 6673 406 0.31 391 3910 376 3760	831 5919 1547 6892 422 0.35 400 4000 4000	1078 6353 2268 7369 453 0.33 478 478 528	1389 6767 2972 7751 475 0.34 530 5133 659	1857 7139 3839 8184 499 0.37 626 861	2338 7407 4365 8443 516 0.36 
rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, pin./in. rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, win./in. rain, pi./in. rain, pi./in.	-155 -1550 2510 31 1.43 -440 1850 46 0.81 258 2580 35 35	103 275 2750 . 23 0.61 61 61 61 245 245 245 245 245 245 245 245 245 245	415 4150 4610 2855 0.16 217 2177 21770 2777 161 0.21 3	675 564 769 5809 372 0.06 330 3300 3300 305 3050 206	766 5804 1271 6673 406 0.31 391 3910 376 3760	831 5919 1547 6892 422 0.35 400 4000 4000 425	1078 6353 2268 7369 453 0.33 478 4780 528	1389 6767 2972 7751 475 0.34 530 5133 659	1857 7139 3839 8184 499 0.37 626 861	2338 7407 4365 8443 516 0.36 
ress, pei real, wiz./in. reas, pei rust, Ié/in. ment, inib/in. reas, pei reas, pei rust, Ib/in. reas, pi rust, Ib/in. reas, pi reas, pi reas, pei rust, µin./in. reas, pei rust, Jb/in.	-1550 2510 31 1.43 .44 185 1850 46 0.81 258 258 35 35	1030 275 2750 ?23 0.61 61 61 245 2450 99 0.65	*150 +61 205 0.16 217 2170 277 2770 161 0.21 3	5644 769 372 0.06 3300 3300 305 3050 206	5804 1271 6673 406 0.31 391 3910 376 3760	5919 1547 6892 422 0.35 400 4000 4000 425	6353 2268 7369 453 0.33 478 478 528	6767 2972 7751 475 0.34 530 5133 659	7139 3839 8184 499 0.37 626 5558 861	7407 4365 8443 516 0.36 578 5649
rein, wir./in. reas, pei rust, li/in. mant, inlb/in. rein, win./in. reas, pei rust, lb/in. reas, pei reas, pei reas, pei reas, pei reas, pei rust, lb/in.	2510 311 1.43 .440 1855 1850 6.61 2580 355 355	275 2750 123 0.61 61 610 245 245 245 245 99 0.65	461 285 0.16 217 2170 2770 2770 161 0.21 3	769 5809 372 0.06 3300 3300 305 3050 206	12/1 6673 406 0.31 391 3910 376 3760	1547 6892 422 0.35 400 4000 425	2266 7369 453 0.33 478 4780 526	2972 7751 475 0.34 530 5133 659	3839 8184 499 0.37 626 5558 861	4365 8443 516 0.36 5649
runs, jui runs, lid/in. runs, inlb/in. runs, pai runs, pai runs, pai runs, pai runs, lb/in. runs, inlb/in. runs, pip./in. runs, pai runs, pai runs, pai	2510 31 1.43 .440 1850 46 0.61 2580 355 355	2750 123 0.61 61 610 245 2455 2455 99 0.65 371 2710	285 0.16 217 2170 2170 2770 161 0.21 3	372 0.06 330 3300 3050 206	406 0.31 391 3910 376 3760	422 0.35 400 425	453 0.33 478 4780 526	475 0.34 530 5133 659	499 0.37 626 5558	516 0.36 5649
rain, µin./in. rain, µin./in. rain, µin./in. rass, psi rass, psi rass, psi rass, psi rain, µin./in. rass, psi rass, psi r	1.43 -44 -440 1850 1850 0.81 258 2580 355	0.61 61 610 245 250 99 0.65 371	0.16 217 2170 2170 2170 2170 161 0.21 3	0.06 3300 305 3050 206	0.31 391 3910 376 3760	0.35 400 4000 425	0.33 478 4780 526	0.34 530 5133 659	626 5558	0.36 078 5649
rain, µin./in. rean, pei rain, µin./in. rase, pei rust, 1b/in. ment, in1b/in. rean, µin./in. rean, µin./in. reas, pei rust, 12/in.	-44 -340 1850 46 0.81 258 2580 35	61 610 245 2450 99 0.65 371	217 2170 277 2770 161 0.21 3	330 3300 305 3050 206	391 3910 376 3760	400 4000 425	478 4780 526	530 5133 659	626 5558 861	078 5649
reas, pei reis, µin./in. ress, pei rest, ib/in. sent, inlb/in. rein, µis./in. ress, pei ress, pei ress, pei ress, pei	-440 1850 1850 46 0.81 258 2580 35	610 245 245 99 0.65 371	2170 277 2770 161 0.21 3	3300 305 3050 206	3910 376 3760	4000	4780 526	51 <b>33</b> 659	5558 861	5649
rain, µin./in. rass, psi rust, lb/in. ment, inlb/in. rusn, pie./in. rusn, psi rusn, psi rust, lb/in.	185 1850 46 0.81 258 2580 35	245 2450 99 0.65 371	277 2770 161 0.21 3	305 3050 206	376 3760	425	528	659	861	
rase, pai rust, lb/in. ment, inlb/in. rean, μin./in. rean, μin./in. reas, pai reas, pai rust, lb/in.	1850 46 0.81 258 2580 35	2*50 99 0.65 371	2770 161 0.21 3	3050	3760				001	1106
rut, in1b/in. ruin, µin./in. ruin, µin./in. ruin, µin./in. russ, psi rut, 1b/in.	0.81 258 2580 35	0.65 371	0.21 3	0.00	3.0	4270	7/47	2010	59/1 275	6403
rain, µin./in. rasa, pai rain, µin./in. rasa, pai ruct, lb/in.	258 2580 35	371		-0.03	-0.05	0.09	0.12	0.18	0.15	0.27
resa, pai rein, µin./in. ress, psi ruct, 16/in.	2580 35	3930	436	662	1551	2650	4068	5510	6916	8047
rain, µin./in. rass, psi ruct, 1b/in.	35	2110	4360	5521	6895	7577	8307	8965	4545	10011
ress, psi ruct, 16/in.		70	245	420	1348	2346	3799	5078	6426	7494
ruct, 16/in.	370	700	2450	4200	6734	7413	8164	8787	9343	9783
	. 95	143	221	332	443	487	535	577	614	643
mat, in16/10.	-0.79	-1.06	-0.67	-0.49	-0.06	-0.06	~0.05	-0.00	-0.07	-0.08
rain, µin./in.	262	459	689	1025	2510	3765	5410	7107	8991	10289
ress, psi	2820	4590	5669	6260	7501	8148	8924	9624	10400	10910
rain, µin./in.	-16	66	212	475	2047	2915	4488	6044	7518	8681
ress, psi	-160	560	2120	4750	7247	TIZO	8503	9105	9793	10272
rust, 10/10.	.1.05	-1 28	-1.37	-0.14	-0.09	-0.15	-0.15	-0.15	-0.20	-0.23
	-1.09	-1.30	-1.33	-0,44	-0.05	-0.19	-0.19	-0.29	-0.21	-0.23
s thrust, 1b/in.	39	111	223	269	328	345	389	414	437	454
g thrust, 10/18.	G.43	0.71	0.89	552 0.82	0.71	0.69	175 17.0	0.70	0.69	0.68
			Test A	$-6 \{z = 1 - 3/1\}$	in.)					
main win /in	-90	-26	167	:20	<u></u>	680	812	1057	1254	1703
rass. nal	-900	-260	1670	3900	5529	5658	5885	6116	6719	7088
rain, uin./in.	218	339	485	728	1059	1237	1480	1893	2321	2831
ress, pei	2180	3390	4850	5737	6320	6633	6839	7163	7398	7674
runt, 1b/in.	42	102	212	333	386	399	418	443	461	480
ment, inlk/in.	1.08	1.29	1.12	0.62	0.27	0.34	0.35	0.27	0.23	0.20
rain, µin./in.	.9	148	331	531	880	1159	1456	1937	2367	2675
ruse, 251 main win /to	104	1400	3310	2130	_2002 	_327	-10020	1101	7423 _6A1	604
MARS. NOT	1060	900	80	-90	2000	-3270	-42-	-5142	-5160	-547)
rust. 1b/in.	-37	. 77	110	140	183	201	2	239	264	292
ment, in1b/in.	0.34	-0.20	-1.14	-2.21	-3.17	-3.63	-3.98	-4.25	-4.25	-4.11
rain, pin./in.	262	525	708	934	1389	2529	3592	5162	6144	7240
ress, ysi	2620	5111	5702	6100	6767	7512	8063	8822	9227	2683
rain, µin./in.	-31	-41	123	260	743	1754	2774	4255	5215	6664
ress, ps1	-310	-410	1230	2600	5764	7057	7644	6369	5 9m	9446
went, in1b/in.	+1.03	-1.99	-1.70	-1.16	-0.37	-0.15	-0.15	-0.15	-0.13	-0.08
rein, µin./in.	199	399	487	612	1161	2268	3421	5113	6307	7890
recs, psi	1990	3990	4870	5498	8499	7380	7979	8802	9294	9951
rein, µin./in.	0	68	158	305	1028	2092	1019	4535	5496	6899
ress, pal	0	680	1580	3050	6265	7272	7776	8526	8959	9542
rust, 16/in.	65	152	210	291	415	476	512	563	593	634
ment, inib/in.	-0.70	-1.17	-1.10	-0.90	-0.00	-0.04	-0.07	-0.10	-0.12	-0.14
g thrust, 1b/in.	40	90	161	237	285	300	317	341	363	386
g tarust, 15/in.	70	155	233	308	413	475	512	562	590	628
*	0.97	0.30	0.09	0.77	0.09	0.05	0.04	0.01	0.02	0.01
	<pre>ment, inlb/iu. roin, µin./in. ress, psi rust, ih/in. g thrust, lb/in. g thrust, lb/in. g thrust, lb/in. ress, psi rust, lb/in. g thrust, lb/in. g thrust, lb/in.</pre>	<pre>ment, inlb/in0.79 roin, µin./in. 282 ress, psi 2820 ruin, µin./in16 ress, psi -160 rust, ih/in. 86 ment, inlb/in. 86 ment, inlb/in. 91 g thrust, lb/in. 91 g thrust, lb/in. 91 g thrust, lb/in. 91 g thrust, lb/in. 91 rust, psi -900 rein, µin./in. 218 ress, psi -900 rust, lh/in. 42 ment, inlb/in. 1.06 rust, lb/in. 9 rust, lb/in. 9 rust, lb/in. 37 ment, inlb/in. 0.34 rust, lb/in. 37 ment, inlb/in. 31 ress, psi 2620 rust, µin./in. 199 ress, psi 390 rust, µin./in. 199 ress, psi 90 rust, ib/in. 75 ment, inlb/in. 65 ment, µin./in. 199 ress, psi 90 rust, lb/in. 75 ment, inlb/in. 75 ment, inlb/in. 65 ment, µin./in. 199 ress, psi 90 rust, lb/in. 75 ment, inlb/in. 65 ment, inlb/in. 70 g thrust, lb/in. 40 g thrust, l</pre>	ment, inib/in0.79 -1.06 rein, µin./in. 262 459 reis, psi 2620 4590 reis, psi 2620 4590 reis, psi 260 4590 rust, lb/in166 66 rust, lb/in. 86 171 ment, inib/in1.05 -1.38 g thrust, lb/in. 91 1157 g thrust, lb/in. 91 157 rein, µin./in. 21B 3390 rust, lb/in. 1.08 1.29 rein, µin./in. 106 300 rust, lb/in. 37 77 ment, inib/in. 0.34 -0.20 reis, µin./in. 262 525 reis, µin./in. 262 525 reis, µin./in. 262 525 reis, µin./in. 37 77 ment, inib/in. 75 157 ment, inib/in. 4.03 -1.99 rein, µin./in. 199 339 rein, µin./in. 199 339 rein, µin./in. 65 152 ment, inib/in. 70 155 g thrust, lb/in. 70 155 0.57 0.58	ment, in1b/in0.79 -1.06 -0.67 rain, µin./in. 262 459 689 ress, psi 2820 4590 5669 rust, psi -160 560 2120 rust, lb/in. 86 171 279 ment, inlb/in1.05 -1.38 -1.33 a thrust, lb/in. 91 157 250 c.43 0.71 0.69 rust, lb/in. 91 157 250 c.43 0.71 0.69 rust, lb/in. 91 157 250 c.43 0.71 0.69 rust, lb/in. 91 157 250 rust, lb/in. 92 148 339 4850 rust, lb/in. 1.06 1.29 1.12 rust, nlk/in. 1.06 1.29 1.12 rust, nlk/in. 9 148 331 rust, min./in. 9 148 331 rust, psi 200 80 rust, lb/in. 37 77 110 ment, inlb/in. 0.3k -0.20 -1.14 rust, µin./in. 9 148 331 rust, µin./in. 262 525 798 rust, lb/in. 75 157 256 ment, inlb/in. 75 157 256 ment, inlb/in. 0.3k -0.20 -1.14 rust, µin./in. 199 399 4870 rust, lb/in. 75 157 256 ment, inlb/in. 0 68 1580 rust, lb/in. 75 157 256 ment, inlb/in0.70 -1.17 -1.16 rust, µin./in. 199 399 4870 rust, lb/in. 75 157 256 ment, inlb/in0.70 -1.17 -1.16 rust, µin./in. 199 399 4870 rust, lb/in. 75 157 256 ment, inlb/in0.70 -1.17 -1.16 rust, µin./in. 199 399 4870 rust, lb/in. 75 157 256 ment, inlb/in0.70 -1.17 -1.16 rust, µin./in. 0 68 1580 rust, lb/in. 70 155 231 0.57 0.58 0.69	ment, in1b/in0.79 -1.06 -0.67 -0.49 rain, µin./in. 282 459 689 1025 ress, psi 2820 4590 5669 6260 rust, in./in1.60 560 2120 4750 rust, ib/in. 86 171 279 372 ment, in1b/in1.05 -1.38 -1.33 -0.44 s thrust, 1b/in. 91 157 250 352 g thrust, 1b/in. 91 157 250 352 g thrust, 1b/in. 91 157 250 352 g thrust, 1b/in. 91 157 250 352 ress, psi 0.43 0.71 0.89 0.82 ress, psi 200 -260 1670 300 ress, psi 218 339 4850 5737 rust, in1b/in. 1.06 1.29 1.12 0.62 ress, psi 90 148 331 531 ress, psi 90 1480 3310 5138 rust, in1b/in. 9 148 331 531 ress, psi 90 1480 3310 5138 rust, in1b/in. 0.34 -0.20 -1.14 -2.21 rust, in1b/in. 0.34 -0.20 -1.14 -2.21 ress, psi 260 500 80 -980 ress, psi 2660 5111 5702 6100 ress, psi 2660 5111 5702 6100 ress, psi 2660 5111 5702 6100 ress, psi 260 5111 5702 6100 ress, psi 2660 5111 5702 6100 ress, psi 2660 5111 5702 6100 ress, psi 2600 5111 5702 6100 ress, psi 2600 5111 5702 6100 rust, 1b/in. 75 157 256 325 rust, in1b/in310 -410 1230 2600 rust, pin./in. 199 399 487 612 ress, psi 0 669 1580 3050 rust, ib/in. 75 157 256 325 rust, in1b/in0.70 -1.16 -0.90 rust, ib/in. 75 157 256 325 rust, in1b/in0.70 -1.17 -1.16 -0.90 rust, ib/in. 10 512 210 291 ment, in1b/in0.70 -1.17 -1.16 -0.90 rust, ib/in. 65 152 210 291 ment, in1b/in0.70 -1.17 -1.16 -0.90 rust, ib/in. 10 510 3050 3050 3050 3050 3050 3050 30	ment, inlb/in0.79 -1.06 -0.67 -0.49 -0.06 rain, µin./in. 262 459 639 1025 2510 reas, psi 2820 4590 5669 6260 7500 reas, psi 2820 4590 5669 6260 7500 reas, psi -160 560 2120 4750 7247 reas, psi -1.05 -1.38 -1.33 -0.44 -0.09 s thrust, lb/in. 91 157 250 352 461 0.43 0.71 0.69 0.62 0.71 Test, µin./in. 91 157 250 352 461 0.43 0.71 0.69 0.62 0.71 reas, psi -000 -260 1670 3300 5529 reas, psi -200 -260 1670 3300 5529 reas, psi -200 -260 1670 3300 5529 reas, psi -218 339 485 728 1059 reas, psi -218 339 485 728 1059 reas, psi -200 -260 1670 3300 5529 reas, psi -218 339 485 728 1059 reas, psi -200 -260 1670 3300 5529 reas, psi -218 330 4850 5737 6320 reas, psi -218 330 4850 5737 6320 reas, psi -200 -260 1670 3300 5529 reas, psi -218 330 4850 5737 6320 reas, psi -200 -260 1670 3300 5529 reas, psi -200 -260 1670 3300 5529 reas, psi -200 -260 1670 3300 5529 reas, psi -200 -260 1670 3300 529 reas, psi -200 -260 1670 3300 529 reas, psi -200 -260 1670 3300 529 reas, psi -200 -200 1.12 0.62 0.27 reas, psi -200 -260 120 -200 reas, psi -200 1.12 0.62 0.27 reas, psi -200 1.12 0.62 0.27 reas, psi -310 440 3310 5138 .005 reas, psi -1060 900 80 -980 -229 reas, psi -210 -114 -2.21 -5.17 rean, µin./in31 44 123 260 7/3 reas, psi -210 -114 -0.37 reas, psi -210 -114 -0.37 reas, psi -210 -116 -0.37 reas, psi -210 -116 -0.37 reas, psi -210 -116 -0.37 reas, psi -199 -1.70 -1.16 -0.37 reas, psi -199 -399 487 519 50 3050 6265 reas, psi -199 -1.70 -1.16 -0.37 reas, psi -199 -399 487 519 54 549 reas, psi -199 -399 487 519 54 52 50 510 268 reas, psi -199 -399 487 519 54 52 50 510 268 reas, psi -199 -399 487 519 54 52 50 91 415 mant, inlb/in0.70 -1.17 -1.16 -0.90 -0.08 s thrust, lb/in. 40 90 161 237 285 s thrust, lb/in. 70 155 233 308 413 0.57 0.	ment, in1b/in0.79 -1.06 -0.67 -0.49 -0.06 -0.06 rein, µin./in. 2820 4599 689 1025 3510 3765 rese, psi 2820 4599 5669 6250 7501 8148 rese, psi -160 560 2120 4750 7747 7720 rese, psi -105 -1.38 -1.33 -0.44 -0.09 -0.15 s thrust, 1b/in. 39 111 223 289 328 345 s thrust, 1b/in. 91 157 250 332 461 502 G.83 0.71 0.68 0.72 0.68 0.71 0.69 rese, psi -900 -260 167 330 619 663 rese, psi -900 -260 1670 3300 5529 5656 rese, psi 2100 3390 4850 5737 6320 6633 rese, psi 2100 3390 4850 5737 6320 6633 rese, psi 2100 3390 4850 5737 6320 6633 rese, psi 90 1480 3310 5138 880 1159 rese, psi 90 1480 3310 5138 800 6496 rese, psi 90 1480 3310 5138 800 6496 rese, psi 1060 900 8 -980 -2290 -3270 rese, psi 1060 900 80 -980 -2290 -3270 rese, psi 290 1480 3310 5138 800 1159 rese, psi 200 1480 3310 5138 800 6496 rese, psi 200 180 -980 -2290 -3270 rese, psi 200 1800 300 500 -980 -2290 -3270 rese, psi 200 1800 300 500 -980 -2290 -3270 rese, psi 200 1800 300 500 -980 -2290 -3270 rese, psi 200 1800 3310 5138 201 -0.27 rese, psi 200 1800 3310 5138 201 -0.27 rese, psi 200 1800 3300 500 -980 -2290 -3270 rese, psi 200 1800 1230 2600 5764 7057 rese, psi -310 -410 1230 2600 7657 7512 rese, psi -310 -410 1230 2600 7657 7512 rese, psi 10/in. 75 157 256 325 411 474 wese, psi 10/in. 75 152 210 231 415 476 rese, psi 10/in. 70 155 233 308 413 475 0.57 0.56 0.69 0.77 0.69 0.63	ment, inlb/in0.79 -1.06 -0.67 -0.49 -0.06 -0.06 -0.06 -0.05 rein, win./in. 262 459 609 1025 7510 3765 7410 rein, win./in16 66 212 475 7517 7517 7730 6503 rein, win./in16 660 2120 4750 7747 7730 6503 rein, win./in105 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 s thrust, lb/in. 39 111 223 209 326 345 345 s thrust, lb/in. 39 111 223 209 326 345 345 s thrust, lb/in. 39 111 223 209 326 345 345 s thrust, lb/in. 39 111 223 209 0.68 0.77 0.69 0.71 c.83 0.71 0.89 0.68 0.77 0.69 0.71 rein, win./in90 -260 167 3300 5529 5658 5685 reas, psi -900 -260 1670 3300 5529 5658 5685 reas, psi -900 -260 1670 3300 5529 5658 5685 reas, psi -106 1.29 1.22 122 333 366 339 418 reas, psi -106 1.29 1.22 0.68 0.77 0.34 0.35 reas, psi -106 50 8 -0.62 0.77 0.34 0.35 reas, psi -112 0.68 0.62 0.77 0.34 0.35 reas, psi -112 0.68 0.62 -229 -327 -425 reas, psi -112 0.68 0.62 -229 -327 -425 reas, psi -112 0.68 0.62 -229 -327 -425 reas, psi -112 0.68 0.68 -207 0.34 0.35 reas, psi -112 0.68 0.68 -207 0.34 0.35 reas, psi -112 0.68 0.77 0.34 0.35 reas, psi -112 0.68 0.77 0.34 0.35 reas, psi -112 0.68 -229 -327 -425 reas, psi -106 90 8 -96 -229 -327 -425 reas, psi -112 0.60 767 7512 8563 reas, psi -110 140 183 201 2 reas, psi -110 140 183 201 2 reas, psi -110 140 1230 2600 5764 7057 7512 8563 reas, psi -310 4410 1230 2600 5764 7057 7512 8563 reas, psi -310 4410 1230 2600 5764 7057 7512 8563 reas, psi -0.20 -1.14 -2.21 -5.17 -3.61 -3.94 reas, psi -0.30 -410 1230 2600 5764 7057 7512 8563 reas, psi -0.20 -1.14 -2.21 -5.17 -3.61 -3.94 reas, psi -0.20 -1.16 -0.37 -0.15 -0.15 reas, psi -0.19 -1.07 -1.16 -0.37 -0.15 -0.15 reas, psi -0.19 -0.70 -1.17 -1.16 -0.37 -0.15 -0.15 reas, psi -0.70 -1.17 -1.16 -0.30 -0.00 -0.00 -0.00 -0.07 reas, psi -0.70 -1.17 -1.16 -0.30 -0.00 -0.00 -0.00 -0.07 reas, psi -0.58 0.09 0.77 0.69 0.65 0.65 0.65 reas, psi -0.70 -0.58 0.09 0.77 0.69 0.65 0.65 0.65	ment, inlb/in0.79 -1.06 -0.67 -0.49 -0.06 -0.06 -0.05 -0.05 -0.06 rein, min./in. 262 459 5669 6260 701 8148 8024 9624 rein, min./in16 66 212 475 2747 2755 4488 6044 9624 9624 rein, min./in160 560 2120 4750 7747 7720 8050 9185 runt, ib/in165 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 -0.15 runt, ib/in105 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 -0.15 s thrust, ib/in105 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 -0.15 s thrust, ib/in105 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 -0.15 runt, ib/in105 -1.38 -1.33 -0.44 -0.09 -0.15 -0.15 -0.15 s thrust, ib/in107 -0.89 0.82 0.71 0.69 0.82 0.71 0.69 0.71 0.70 runt, ib/in20 -26 167 370 619 663 812 1057 runt, ib/in218 339 465 726 1059 1237 1480 1893 runt, ib/in218 339 465 726 1059 1237 1480 1893 runt, ib/in106 1.29 1.12 0.62 0.27 0.34 0.35 0.27 runt, ib/in106 1.29 1.12 0.62 0.27 0.34 0.35 0.27 runt, ib/in106 1.29 1.12 0.62 0.27 0.34 0.35 0.27 runt, ib/in106 90 868 -229 -327 -425 -532 runt, in1k/in106 90 868 -229 -327 -425 -532 runt, in1k/in106 90 8 -58 -229 -327 -425 -532 runt, in1k/in. 0.3 -0.20 -1.14 -2.21 -3.17 -3.61 -2. 229 -327 -425 -532 runt, in1k/in. 0.3 -0.20 -1.14 -2.21 -3.17 -3.61 -2. 229 -327 -425 -532 runt, in1k/in. 0.3 -0.20 -1.14 -2.21 -5.17 -3.61 -2. 219 -327 -425 -532 runt, in1k/in. 0.3 +0.20 -1.16 -0.37 -0.15 -0.15 -0.15 -0.15 runt, in1k/in31 -41 123 260 744 1754 274 4.255 runt, in1k/in31 -41 123 260 744 1754 274 4.255 runt, in1k/in31 -41 123 260 744 1754 2764 4.259 runt, ik/in31 -41 123 260 744 1754 2764 4.259 runt, ik/in31 -41 123 260 744 1754 2764 4.255 runt, ik/in0.70 -1.17 -1.16 -0.37 -0.15 -0.15 -0.15 -0.15 runt, ik/in0.70 -1.17 -1.16 -0.37 -0.15 -0.15 -0.15 runt, ik/in0.70 -1.17 -1.16 -0.37 -0.15 -0.15 -0.15 runt, ik/in0 660 1560 1	ment, inlh/in0.79 -1.06 -0.67 -0.49 -0.06 -0.06 -0.05 -0.06 -0.07 rean, in./in. 262 459 569 569 569 526 750. 8148 Break 5624 10400 rean, in./in16 66 212 k75 2747 2915 4488 6644 7518 rean, in./in16 66 212 k75 2747 2915 4488 6644 7518 rean, in./in16 66 212 k75 2747 2915 4488 6644 7518 rean, in./in16 10 660 2120 k750 7747 7760 893 9185 9793 rean, inlb/in105 -1.38 -1.33 -0.44 0.099 -0.15 -0.15 -0.15 -0.21 etails, in1b/in1.05 -1.38 -1.33 -0.44 0.099 -0.15 -0.15 -0.25 -0.21 s thrust, lb/in. 39 111 223 289 326 345 349 40.071 0.69 0.69 0.71 0.69 0.69 0.71 0.69 0.69 0.71 0.69 0.71 0.69 0.69 0.71 0.69 0.71 0.70 0.69 rean, in./in. 91 157 250 352 461 502 551 594 635 rean, in./in. 91 157 250 352 461 502 551 594 635 rean, in./in. 218 339 445 778 1079 1237 1480 1893 2221 rean, in./in. 218 339 445 778 1079 1237 1480 1893 2221 rean, in./in. 218 339 445 778 1079 1237 1480 1893 2221 rean, in./in. 1.06 1.29 1.12 0.62 0.27 0.34 0.35 0.27 0.23 rean, in./in. 9 148 331 531 880 1159 1496 143 461 443 461 rean, in./in. 9 148 331 531 880 1159 1496 143 461 443 461 rean, in./in. 1.06 1.29 1.12 0.62 0.27 0.34 0.27 0.23 rean, in./in. 1.06 1.29 1.12 0.62 0.27 0.34 0.23 70 4.68 6620 7187 7423 rean, in./in. 1.06 1.29 1.12 0.62 0.27 0.34 0.23 0.257 0.23 29 264 rean, in./in. 1.06 1.29 1.12 0.60 2.0.27 0.34 0.23 0.257 0.23 rean, in./in. 1.06 1.29 1.12 0.62 0.27 0.34 0.25 0.27 0.23 rean, in./in. 1.06 1.20 1.10 1.10 1.20 0.27 0.35 0.27 0.23 rean, in./in. 1.06 1.20 1.13 1.20 0.62 0.27 0.34 0.53 0.62 0.66 6620 7187 7423 rean, in./in. 1.06 1.20 1.1 700 163 2.20 1.20 0.27 0.35 0.25 0.35 0.35 0.35 0.35 0.35 0.35 0.35 0.3

Table 5.4 Strain, Stress, Thrust, Moment, and Deflection; Tests B-1A, B-1B, B-2, B-3, B-4, B-5

						Over	pressure. p	ui				
Gage	Measurement		100	150		250	300	115	350	400		300
1	Strain, µin./in.	177	322	257	<u>المار مارد</u> بلبل	- <u>v 28.7</u> 16	-80					
la	Stress, psi. Strein, uin./in.	1.70	3220	2570	1440 814	160	-800 1k99					
3-10	Stress, psi	1380	2760	4970	8140	11187	11861					
1-16	Moment, in1b/in.	-0.14	-0.16	0.84	2.36	305 4.04	420					
3	Strain, µin./in.	-49	-81	-65	-65	-32	-16					
34	Strain, µin./in.	-490	-610	-650 276	-650 385	-320 491	-160 بلبلخ					
3-3 <b>4</b>	Streas, psi Thrust, lb/in	1060 19	1850 34	27 <b>8</b> 0 69	3850 104	4910	5440					
•••	Moment, in -1b/in-	0.55	0.94	1.21	1.58	1.84	1.97					
2	Strain, µin./in. Stress, psi	112 1120	205 2050	280	410 4100	560 5600	672 6720					
28	Strain, µin./in.	105	134	239	314	388	448					
2-28	Thrust, 1b/in.	'n	110	169	235	308	364					
L	Strein uin /in	-0.02	-0.25	-C.14	-0.34	-0.61	-0.79					
ч 1.	Stress, psi	1720	3100	4490	5870	7420	8460					
4&	Strain, µin./in. Stress, psi	40	79 790	158 1 <b>58</b> 0	211 2110	264 2649	290 2900					
4-4 <b>e</b>	Thrust, 1b/in. Moment. in1b/in.	-0-69 Aul.o-	126	197	259	327	369					
DC1	Deflection, in.	0.013	0.019	0.022	0.026	0.035	0.035					
DC2	Deflection, in.	0 61	0.003	0.006	0.006	0.009	0.009					
2-2a14-4a	Avg thrust, 1b/in.	70	118	183	247	318	367					
	q	0.87	0.97	0.86	0.84 	∩.84	0.81					
1	Strain, uin /in	-76	-152	_2 <b>8</b> 1	1785 1-18 (2	_62R	-810	- Calus				
-	Stress, psi	-760	-1520	-2610	-4340	-6280	-8100	-9450				
18	Stress, psi	243 ≥430	5270 5270	8890	11402	1705						
1-1 <b>a</b>	Thrust, 1b/in. Moment. in1b/in.	54 1.12	122 2.39	19 <b>8</b> 4,12	264 5,84	311 7.15						
3	Strain, µin./in.	-35	-42	-49	-69	-97.	-124	-131				
Re.	Stress, psi Strein, uin./in.	-350	-420	-490	-690	-970 515	-1240 611	-1310 625				
	Stress, poi	1090	2120	3160	4530	5150	6110	6250				
5-54	Moment, in1b/in.	0.51	0.89	1.29	1.84	2.15	2.59	2.66				
2	Stre :, µin./in.	100	207	331	446	577	700	706				
24	Strein, µin./in.	65	123	212	294	388	465	482				
2-2	Stress, poi Thrust, lb/in.	650 54	1230 107	2120 176	2940 2940	3880 314	4650	387				
	Moment, in1b/in.	-0.12	-0.30	-0.42	-0.54	-0.67	-0.83	-0.80				
4	Strain, Lin./in. Straat, nai	115	257 2576	379	494 Hoho	629 6290	744 7440	751 7510				
4 <b>a.</b>	Strain, µin./in.	41	116	185	247	316	384	391				
4-44	Thrust, 1b/in.	51	121	183	241	307	367	371			-	
1001	Noment, in1b/in.	-0.26	-0.50	-0.65	-0.87	-1.10	-1.27 0.044	-1.87				
DC2	Deflection, in.	0.003	0.004	0.007	0.013	0.016	0.016	0.016				
1-1a:3-3a 2-2a:4-4a	Avg thrust, lb/in. Avg thrust, lb/in.	39 53	89 114	143	195 241	311	373	379				
-	9	0.74	0. <b>78</b>	0.79	0,81	0.72	••	••				N (1
	<b>O</b> tomatic 1.4 - 14 -	150		<b>1</b>	hant 11-5 (Z -	7/16 in.)	0#B		140	•		-
1	Stress, psi	1500	3260	4210	4340	3650	2580		1290	· · ŏ.	-1410	~ <b>369</b> 0
16	Strain, µin./in. Stress. psi	72 720	••	••			••		••	**		
1-18	Thrust, 1b/in.	72	**		••	••					••	
٦	Strain, uin./in.	-67	-108	-83	-61	-62	-71		-71	-79	-79	-117
2.	Stress, psi	-670	-1080	-830	-670	-620	-710		-710	-790	-790	-1170
<b>.</b>	Stress, pel	8120	2990	4140	5030	5850	6730		7890	9080	10190	11074
3-34	Tarust, 15/18. Moment, in15/in.	0.98	1.43	1.75	2.01	2.26	190	·	3.03	3.48	3.87	349
5	Strain, µin./in.	118	190	262	338	445	545		658	777	894	1040
24	Strein, uin./in.	1180 36	1900 98	2620	3360	4450 383	5450 417		6580 515	60	89N0 736	10400
2-24	Stress, pol	360	9 NO	146	2320	3830	4170	x.	5150	éithe has	7360	10110
	Moment, in1b/in.	-0.89	-0.32	-0.34	-0.37	-0.43	-0.45		-0.90	-0.96	-4.56	-0.10
4	Strain, Min./in.	.99 800	165	264	343	475	587 9870		734	861 8610	1088	1663
48	Birein, uin /in.	6	103	176	239	380	397		483	569	699	1066
4-44	Shrust, 15/12	54	67	243	109	158	380		396	465	346	10000
- 1.01	Minnest, in1%/in.	-0.11	-0.29	-0.31	-0.37	-0.55	-0.67		-0,55	-1.03	+1.88	-0.44
DCS	Deflection, in.	0.001	0.002	0.003	0.001	0.005	0.008		0.010	0.013	0.035	0.018
1-1413-34 2-2410-64	Ave thrust, 10/18.	60 52	<b>Q</b> 1	141	387	84k			380	Mo		711
		1.55										

(Continued)

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Table 5.4 (Concluded)

<i>_</i>						Overs	ressure, ps	1				
	Handlingsont		100	 	$\frac{200}{100}$	<u>- 250</u> 7/8)	300			400	450	
1 *	Strain, win./in.	65	158	201	208	201	187		158	129	100	43
14	Stress, pei Strain, µin./in.	650 22	1 <b>58</b> 0 111	2010 بلبا2	20 <b>8</b> 0 377	2010 544	1 <b>87</b> 0 710		1580 899	1290 1071	1000 1270	430 1457
1-14	Stress, psi Thrust, 15/1n.	220 28	1115 87	2440 145	3770 190	242 242	7100 292		8990 344	10710 <b>390</b>	11440 439	11941 466
•	Noment, in1b/in.	-0.15	-0.17	0.15	0.60	1.21	1.85		<b>S</b> .61	3.32	3.94 32	4,40
3	Stress, pei	-510	-320	-60	60	130	60		, or an	-120	-320 -320	-450
3 <b>8</b>	Stress, pei	1640	2720	3690	4510	5630	6530		7870	910	10400	11149
5-38	Moment, in1b/in.	57 0.76	1.07	1.32	1.57	1.94	2.28		2.77	3.25	3.77	4.21
2	Strain, µin./in. Stragg. pai	187 1870	315 3150	460 4600	581 5810	737 7370	862 8620		1008 10080	1152 11133	1321 11581	1923 12145
25	Strain, µin./in.	14	75	136	186	254 2540	308 3080		351 3510	455	573 5730	1070
2-2a	Thrust, 1b/in.	65	127	194	249	322	380		442	521 -2.43	600 •2.22	764
4	Strain, µin./in.	101	191	303	397	529	633		759	880	1044	1764
4.	ðtress, psi Strein, μin./in.	1010 34	1910 103	3030 168	3970 229	5290 306	6330 367		7590	8800 516	10k40 611	12075
la-la	Stress, pei Thrust, 1b/in.	340 հե	1030 96	1680 153	2290 203	3060 271	3670 325		i4430 391	5160 մե5հ	6110 5 <b>3</b> 8	9280 741
-	Moment, in1b/in.	-0.24	-0.31	-0.48	-0.59	-0.79	-0.94		-1.11	-1.28	-1.52	-0.82
DC2	Deflection, in.	0.001	0.009	0.002	0.014	0.007	0.009		0.013	0.015	0.017	0.019
1-1813-38 2-2814-48	Avg thrust, 15/1n. Avg thrust, 15/1n.	33 55	63 112	174	226	215	353		417	486	569	753
	q	0.60	0.74	0.75 Test	0.75 B=3 (Z = 1	-3/4 in.)	0.72		0.72	0.70	0.67	0.35
1	Strain, µin./in.	7	48	75	82	82	75		65	61	41	27
la	Stress, psi Strain, µin./in.	70 103	480 192	750	820 378	820 494	750 603		650 722	610 819	410 944	270
1-1 <b>6</b>	Stress, psi Thrust, 1b/in.	1030 36	1920 78	2740 113	3780 149	187	6030 220		7220	8190 286	9440 320	10550 352
`•	Noment, in1b/in.	44. الأنار	0.51	0.70	1.04	1.45 _h1	1.86 -6)		2.31	2.67	3.18	3.62
3	Stress, psi	-440	-440	-350	-350	-410	-610		-830	-1040	-1300	-1480
34	Stress, psi	1530	2700	3640	4520	5610	6700		7730	8990	10330	11144
3-38	Moment, in1b/in.	0.69	1.11	1.40	1.71	2.12	2.57		3.01	3.53	#100 532	4.57
2	Strein, µin./in. Stress, psi	158 1580	268 2680	365 3650	461 4610	574 5740	671 6710		768 7680	877 8770	9920 9920	1183
28.	Strain, µin./in. Stress, psi	10	69 690	121 1210	1 <b>78</b> 17 <b>8</b> 0	243	3020		374 3740	446 4460	515 5150	679 6790
2-26	Thrust, lb/in. Moment. inlb/in.	55 +0.52	110	1 <b>58</b> -0. <b>86</b>	20 <b>8</b> -1.00	26( -1 17	316 -1.30		371	430	490 •1.68	612 -1.68
4	Strain, µin./in.	218	371	502	625	775	897		1040	1153	1209	1179
4a.	Strein, µin./in.	-32	3710 8	5020	6250 58	100	142		187	261	409	736
4-44	Streas, psi Thrust, 1b/in.	-320 60	-80 118	180 169	580 222	1000- 2 <b>8</b> 4	1420 338		1870 399	2610 459	4090	<b>736</b> 0 619
<b>10</b> 11	Moment, in1b/in.	-0.85	-1.33	-1.70	~2,00 0,013	-2.38 6-016	-2.66 a.016		-3.00 0.023	+3.12 0.096	•2.71 0.010	-1,440 21,123-5
DC2	Deflection, in.	106.0	0.007	0.010	0.011	0.012	0.014		0.016	0.019	0.021	0.0%
2-2214-48	Avg thrust, 10/in.	58	114	164	215	275	327		305	445. 0.61	506	611
	۲.	0.02	0.01	Seet	3-4 (Z + 2	-5/8 in.)	0.00		V. G8	0.04	0.04	0.20
1	Strain, win./in.	-49	-36	-4	29	60	78		98	111	122	111
la	Strain, win./in.	129	- 100 215	305	399	506	610		735	854 854	- 980	4 + 457 - 1111 - 112-000
l-la	Thrust, 1b/in.	1190	2150 58	37390 9 <b>6</b>	139	184	224		271	314	361	197
٦	Strain. uin./in.	0.63 •130	0.665 -177	1.09 -194	- 1.30	1.57 -212	1.67 -297		-252		-303	5.52 •125
- 1a	Stress, pei Strein, uin /in.	-1300	-1770	-1960	-2030	+2120 552	-2270		-2520	-2790	- 3030 970	- 32%) 1054
1-14	Stress, pai Thrust, lb/in.	1910	3000	3910	4680	5520 110	6310		7360	8460	9700 217	107946
J-3=	Moment, in1b/in.	1.13	1.68	2.06	£.36	2.69	3.08		3.48	3.96	4,68	4.66
2	Streis, µin./in. Stress, pei	176	276 2760	3700	453 4530	555 5550	6510		765 7650	8730	1005 10080	1108
24	Strein, µin./in. Stress, psi	-22	26 260	91 9 <b>7</b> 0	178	265 2650	344 3440		640 6400	537 5370	645 6450	9460 9460
2- <b>3</b> 8	Thrust, 1b/in. Moment, in1b/in.	49 -0.69	-0. <b>8</b> 9	158 =0.96	205 =0.97	266 -1.02	323 -1.08		392	458 -1.18	537 -1.20	-0.57
4	Strein, pin./in.	219	361	ligh hole	617	761	890		1042	1189	1337	1480
ka,	Strain, µin./in.	-90	-43	+17	15	1010	93		137	193	C83	1971 122 1.884
le-leg	Thrust, 1b/in.	55	203	175	205	265	319		303	447	514	305
DC1	Deflection. is.	-0.95	-1,42 0,014	-1.00	-#-12 0.020	44.49 9.0 <b>83</b>	0.026		+3+19 +3+19	-3.47 0.034	0.039	•€-94 0.044
SCH 1-1-1-	Deflection, in. Ave thrust, ih/im.	0.006	0.008	0.010	0.012	0.014	0.016		0.718	0.080 249	190.0 190.0	0.023 317
2-2015-40	Avg thrust, 1b/in.	52 0.44	101 0.49	154	205	2 <b>66</b> 0.55	321 0.55		388	453 0.55	586 9.55	626 0.51
	-											

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Table 5.5 Strain, Streas, Thrust, and Humant; Tests 5-6, 5-7, 5-8, 5-9, 5-10

						Overtress	NITE. DEL				
	Heartment		_ <u></u>	150	<b>b</b> 6 (7 = 0	2 <u>50</u>		350		- 550	
1	Strain, µin./in.	-239	207	414	233	-65	-174	-496			
ia.	Stress, psi Strein, µin./in.	-2390 349	2070	*140 232	2330 791	-650 1385	-1760 1699	-4980 2665	6471		
1-10	Stress, psi	3490	24 ) 7	2320	7910	11758	12047	12069	13050		
	Noment, in -1b/in.	2.07	0.13	-0.64	1.96	4.73	5.36	6.30	••		
3	Strais, µis./is. Strass. cai	-5k -5k0	123	508 208	216	189	185	162	131		
3a	Strain, µin./in.	144	220	275	315	502	613	743	785		
3-3 <b>a</b>	Thrust, 1b/ia.	1440	111	2750	3720 191	5020 225	6130 259	7730	7050 296		
	Nomest, in1b/in.	0.70	0.34	0.24	0.55	1.10	1.51	1.96	2.30		
2	Strein, µin./in. Stress, pei	257 2570	383 3830	494 4940	646 6460	877 8770	998 9980	1167 11172	1230 11334		
2 <b>6</b>	Strein, µin./in. Strees. nei	-45 -450	13	108	240	336 3360	399 1990	463 6630	527		
2- <b>2</b> 2	Thrust, 1b/in.	69	129	196	285	394	454	528	565		
	Strain, un./in.	239	301	-1.50	-13	• 4 90 407	720	-4.43 Ren	34		
- h-	Stress, pei	2390	3010	3710	4460	5970	7290	8230	8600		
*8	Stress, pri	-160	630	1580	2220	200	. <b>36</b> 40	479	5230		
4-4 <b>6</b>	Thrust, 1b/in. Homest, in1b/in.	-0.90	118 +0.84	-0.75	217 -0.79	<b>207</b> +1.10	355	417 -1.28	496 -1.26		
1-1a:3-3a	Avg thrust, 1b/in.	33	129	184	262	321	355	387	••		
2- <b>2619-96</b>	Avg thrust, 1b/in.	71 0. <b>46</b>	1.04	184	253 1.04	341. 0.96	405	473	511		
	•			Test	-7 (2 - 7/1	<u>6 18.)</u>					
1	Strais, µis./is.	-145	294	641	716	81	558	NON.		368	277
la.	Strain, pin./in.	-1450	175	111	7160	6#10 455	5580 671	876	1068	3000 1345	2770
1-1 <b>8</b>	Stress, pei Thrust, 1b/is.	2470	1750	1110	2470 313	4550	6710 399	8100	10620	11639	12006
	Noment, in1b/in.	1.36	-0.42	-1.87	+1.65	-0.65	0.40	1.39	2.61	2.96	3.53
3	Strein, µin./in. Stress. pai	510 51	51 <b>9</b> 0 51 <b>9</b>	367	149 1490	502 5020	530 5300	590 5900	606 6060	652 6520	700
38	Strais, mis./is.	145	154	179	218	273	350	353	466	539	599
3-34	Devet, 10/15	54	121	177	817	858 858	206	319	346	301	622
• ·	Numera ut /in	0.44	-0.23	-0,66	-0.81	-0.61	-0.63	-0.69	-0.49	-0.40	-0.36
	Street, pei	2590	3320	4130	5680	7080	9110	9960	11078	11238	11691
<b>A</b> .	Birein, #18./18. Birees, pei	-80	50 500	136	2110 2110	29N0	3940	inter a	2370	61A 61A0	701
2-24	Thrust, lb/in. Moment, in1b/in.	-0.9h	-0.99	176	- 251	394	1.82	468 	539	609	654
•	Strain, pin./in.	896	310	1423	586	695	893	1007	116	1338	3 Marine
-	Stress, pti Stress, uin./in.	2500 30	3100	1250	5860	6950	8950	10010	11175	11612	11984
	Stress, pel	100	1090	1860	2700	3540	4370	5960	6090	1010	-
	Numeri, inlb/in.	-0.87	-0.75	-0.83	-0.90	-1-10	-1.4	-1.69	-0.92	-1.71	-0.65
1-1413-34	Ave threat, 12/1n.		137	811	865	304	2	31	440	487	129
	d and and and a relation of the second se	0.52	1.06	5-73	1.0	0.91	0.00	0.79	0.75	0.7	0.78
_			-	3unt.	H (1 + 7/8	ليتل الم					
1	Burnin, pin./18. Street, pei	-97	137	298 2980	369	3460	ying	356	349 3490	319	
	Strain, pin./in.	261	851 8510		-	683	1	2		-	
<b>1-16</b>	Derust, lb/in.	3	114	808		30	378				**
	Mania uta /ia.		444	0.30	9.87 - 1199	1947	****	<b>4</b> -3 <b>4</b>			
	Stress, pul	- 100	7750	9790	ing	11700	**	**		••	
<b>.</b>	Breis, pei		-8770	-4170	-1740		<b>**</b>		**	84. 89	-684
3-3 <b>4</b>	Thrust, 1h/1n.	-1.0E	4.75	· 183 ·	-5.77		**	**	**	88 34	816 846
∎ statis	Strain, pin./in.	23	385		559	700		997	1199	1.007	1995
	Street, pei Strein, sin./in.	2180 () 14	3850		5390 297	7000		9970		11430	
h.m.	Stress, pel	140	800	2050	2070	3730	4790	5440	690	6050	
	Humani, in10/1n.	-1.72	-0.Te	-0.79		-1.15	-1.3		-1.71	-1.4	20.9
•	Strain, sin./in.	175	177	330	116 1.14	798	497	11	1	1001	H
	Strain, pin./in.	17	106	199	871	351	The second	NY I		-	71
	Thread, pai Thread, 12/10.	170	119	178	#710 #23	306	100 - 100 -			981	- 7180 (21)
	Nument, inIb/in.	-0.96	-0.9	-0.46	-0.91	-0.45	-0.05	-1.05	-1.19	-1.99	-1.6
	Ang threat, 14/16.	2						in the second se	517	-	
		0.75	0.98	1.0	0.99	***			· · · · ·		

Course & much

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Table 5.5 (Concluded)

							-		1707		
									**-		
1	Strain, Min./M.		34 340	107 1090	1465 14640	189 1890	192	2240	2490 2490	2099 20190	273 2530
34	Bunin, vin./in.	261	443	495	(m	711	66	986	1068	11/18	1201
1-10	Stress, pal. Second. 12/10.	2620 46	1450	4970	681.0 275	7110	331	9980 377	10680	476	11478
<i></i>	Names, in15/2n.	1.8	1.44	1.37	1.61	1.8	2.24	2.51	2.80	3.10	3.41
3	Strain, sin./in.	-49	-9	52	180	172	180	215	258	203	292
•	Strees, pel. Street, sta./ta.	-490	-90 110	990 100	1900	531	617	696	789	890	3980 980
-	Street, pel	2000	3300	3800	1660	5310	6170	6960	7890	8900	9800
3-34	Shraft, Dyln. Hannet, inDyln.	0.95	1.19	140 1.15	190	1.26	1.54	1.69	1.87	2,14	2.42
	Strain, sin./in.	809	379	186	648	770	876	1091	1163	1386	1485
-	Stress, pel	8090	3790	4860	6880	7700	8760	10910	11168	11601	11953
-	Reis, pis-/is-	600	1980		3790	4590	5190	6(70	7130	9060	9110
8-8a	Thrust, 1h/1a.		100	250	327	399	453	552	608	670	717
	Remark, inD/18.	-0.51	-0.04	-0.72	-0.00	-1.09	-1-80	•1.70	• * • 76	*4.67	
•	Strain, sin./in.	<b>2</b> 11	2	467 1.490	646 6460	776	885	1099 30890	1163	11617	1496
-	Strain, sin./in.	41	149	235	307	397	461	551	683	723	949
h h-	Stress, jet	110	1490	2350	3070	3970	461C	5510	579	7850 847	9490 730
	Hannet, inlb/in.	-0.60	-0.83	-0.82	-1.80	-1.33	-1.49	-1.75	-1.85	-1.63	-0.74
1-1013-30	Ang threat, 11/1a.	2	125	348	883	360	295	337	30	130	*53
8-8619-96	any shreet, DV1s.	0.60	0.65	0.70	0.70	0.67	0.66	0.63	0.57	0.65	0.63
				Ins. I	-10 (3 - 3-5	(8. ja.)					
1	Stain. in/in.	-113	-11	35	89	116	134	148	149	284	177
-	Stress, sei	-1130	-670	390	890	1160	1340	1480	1690	1055	1770
<b>16</b>	Birne, 11.	1990	3510	i lato	940	6440	740	4370	9500	10650	11200
3-34	Barast, 14/1a.		. 2	195	805			390	36	406	139
	Manuel, inlb/lin.	1.10	1.47	1.43	1.33	1.07	g. 14	e.ng	#1 [2	3.00	
3	Strain, pin./la. Stran. mi				 	580	300	379	260	670	680
<b>%</b>	Strein, sin./in.			397	190	941		708	618 040	: 989 anim	1080
3-34	Savet, 12/10.	39		114	145	183	815	84	80	- <b>1</b>	356
	Numme, inlb/in.	0.69	1.51	1.56	1.60	1.85	÷ #.12	<b>8.36</b>	1.67	- 3-09	3.30
2	Strain, sin./in.	214	411		646	76	913	2011	1200	1349	3466
	Street, pel.	2040	110	214	210	376		527	689	739	600
	Stress, pel	ko	:350	0845	2900	3760	M20	5870	6890	1196	0.00
2-25	Marant, 14/18.	-0.71	-1.05	-1-15	<b></b>	-1-37	-1.99	+1.98	.1.69	-1.65	-1.8
	Sirein, sis./is.	20A		773	667		979	1086	1200	1418	1537
	Street, pai	8010	4380	5730		1000 - 10000 - 10000 - 1000 - 1000 - 1000 - 1000 - 1000 - 1000 - 1000 -	114	10060		410	4976
an <mark>n</mark> a se g	······································	-130		. <b>36</b> 0	1940	2760	1960	4100		5100	<b>A10</b>
h-la -	Brief, 12/1a.		161	105	2001. 	300		1000 - 10000 - 10000 - 1000 - 1000 - 1000 - 1000 - 1000 - 1000 - 1000 -	2 <b>2 2 2</b> 2	516 7, 79	-2.01
				-4+33	- 3. 1 Mar					1	
And and a second second	Ang shares, 12/14.	5	144	<b>#</b> C	177 1814	36	-37	10	371		
	•	0.51	0.53	0.56	· · · · ·	9. <b>99</b>	0.37	9.56	9.90	0.99	0.39
		• .		1. S.		1.		8			1997 - 1997 -

				- Ti	mble 5.6						
a,	Stress,	Thrust,	Nonest ,	and	Deflection;	Tests	C-1,	c-2,	C-3.	C-4,	C-5

Gase	Man our remark	- 8	40	- 75	55	100	150		Tt. pai	7.0	300	150	100	850	500
						Inst. Co.)	(2.0	<u>11.</u> )							ختف
1	Strain, µin./in. Strass. pei	159 1590	63% 63%0	••	••										
ie	Strain, µin./in.	170	156	71 710	-453 -4530										
1-18	Thrust, 1b/in. Nument, in1b/in.	36 0.00	87 -0.19		**										
3	Streis, sis./is.	262	470	616	739										
36	Strain, pin./in.	-69	-106	-119	-125										
3-3a	Thrust, 10/18.	21	-1080	55	68										
2	Birein, sis./is.	125	-0.23 234	33Å	-0-30 450										
2	Strees, pei Strein, pin./im.	1250	2340 121	3340 205	4500 320										
2-2	Strees, pai Threat, li/in.	400 19	1210	2050 59	3200										
	Noment, in1b/is.	-0.03	-0.05	-0.05 387	-0.35										
ha	Strein, pri Strein, uin./in.	1990	2750	3870	4700										
11a	Stress, pai Thrust, 1b/in.	260	620 37	1100	1530										
-	Monumet, im1b/im.	-0.05	-0.09	-0.11	-0.13										
DC5	Deflection, in.	0.003 0.003	-0.003	0.008 0.003	0.009										
1-1613-36 2-2616-66	Ang threat, 15/18. Ang threat, 15/18.	29 20	36	57	π										
	•	1.45	1.05	••	**	Sent C-A	( <b>7</b> - 3/1	6 <b>1a</b> .)							
1	Strain, sin./in.		591			1087	1604	1907							
1.	Strain, Mis./is.		105			308	495	822							
1-la	Thrust, 15/1n.		14			3000 2.	2 <u>3</u> 1	300							
3	Reals, uis./is.		358			519	686	760						÷	
34	Stress, pai Strein, sis./in.		3580			5190	- 400 -	7600							
3-34	Street, 12/1n.		-1790			-1590	-100	100 86							
•	Manuel, inib/in.		-0.21			-0.27	-0.31	-0.30 648							
•	Stress, pai		1950			JUO	100	0.00							÷
**	Stress, pel Drust, ib/is.		3630			3260	7900	10440							
	Homent, inib/in.		0.0			0.09	0.12	9.14							
•. •.	Wrose, pei		-				8170	10000							a de la composición d
and and a second	Marese, pai		-				7500	1000							
	Mount, in -15/12.		-0.0			-0.08	-0.03	0.0							
9071 907	Deficition, in. Deficition, in.	n de la Neterre	0.005			0.001 -0.001	0.008 -0.001	C.025				salara ang			
1-1413-38 2-8413-38	Ang threat, 10/18. Ang threat, 10/18.		- 67 - 51			97 1403	146	193							
			<b>6.98</b>			. 0.94) Thet C.S	0.9h 12 . 4/2	(9),992 Aitan,}						i.	
<b>1</b>	Mpale, sis./is.		-			981	1984		1639	Lines.	ELO.		8485	2335	8911
18	Macia, pin./in.		115				1940		601	100 A	1319	170	2010	2100	49779 **
1-14	Thrust, 11/10.		200 200 200							34			3W	376	**
3	Persia, sin./in.	gen <sup>sen de</sup> Rosen de la			+ 1 <sup>11</sup>		411		391			973	1108	2005	1399
. 🔊	Stress, pol Strein, nin./in.					3490 835	TO		9530 923			9730 98	11000	1396	13990
3-34	Street, pol		970 31						9900 113	151		9960 817	11,900	13060	1980
4	Reason, in12/10-		49.0+ 49.0+			413.09	-45.03 991		-0.01	0.01 1100	0.05 1407	9.01 1914	0.0% 2011	0.00	0.07
•	Nouse, pet	•				SUNO	TTA		1040		1101	194.90		2000	••
	Street, pel		- P				3300		7740	NOP OF	LULIO	1000			••
	Names, 10 - 10/10.		-0.0			-0.00	-9.30		-0.11	-0.11	-0.12	-0.13		-9.18	••
•	Struce, pal		174	al an An an			-			-	LUN	Junto		19110	
in the second se		11. 11.	ેન્ટ્ર <b>ન્દ્ર</b> ્યુ				200		339			11090	13000	19870	173
	Hannali, Ma13/16.		-4). 61			-4.W	-0.2		-0.04	49. <b>9</b> 7	-4,10	-9-13	-0.14	-0.1	-0.48
	befiertien, in.	•	0.000			0.000	0.00L		406.0 000.C	0.000 0.000	0.013 9.005	0.019	0.085 0.080	0.051 0.013	0.015
1-3413-34 3-845-64	Ang Marust, Maria.					-						1	303		40
	1		1.9			1.48	1.00		1.11	1.06	1.98	0.99	<b>4.</b>	9.95	**

ŵ,

## Table 5.6 (Concluded)

					Overseente. P	1					
			<u>95 100</u>	150		20	300	350	400	470	
			Test C-	<del>2 (Z -</del>	7/16 in.)						
1	Strain, µia./ia.	426	694	886	10	4 112	1180	1193	1217	1266	1290
	Stress, pai	<b>426</b> 0	6940	3650	103	10 1120 50 71	0 11800	11930 1408	12170	12660	12900
18	Streis, MIR./IR.	140	960	2650	46	ñ ni	10180	14080	17980	22450	27170
1-18	Thrust, 1b/1a.	46	<b>87</b>	127	1	SH 20.	242	286	332	386	երի 1141-1
	Humit, 1819/13.	-0.17	-0.64	+0.27	-0.1	.j •0.1	-0.01	0.09	v	0.39	0.70
3	Strain, win./13.	31	107	176	2	57 35	L 443	550	642	779	909 anan
30	Stress, psi.	302	1070	647	8	01 96	1154	1369	1566	1811	2032
	Stress, pel	3020	4800	6470	80	LO 964	11540	13690	15660	18110	20320
3-34	Thrust, 1b/in.	37 0.11	0,15	91.0	0.3	22 0.2	5 0.29	0.33	0.37	0.42	0.45
	minute, sources and	V.11		,				<b>0</b> -			161-0
2	Strain, win./in.	83	158	253	37	72 51 20 515	5 6690	8470	10520	13320	16400
28	Strein, Min./in.	75	133	195	2	70 37.	489	616	763	982	1200
	Stress, pel	750	1330	1950	27	20 373	o 4890 }	6160	7630	9820	12000
2-26	Thrust, 10/18.	17	-0.01	-0.02	-0.0	04 - 0.0	6 -0.07	-0.09	-0.12	-0.14	-0.18
	······································		•		-		• 6he	<b>70</b> 043	077	1207	չերեր
<b>b</b>	Strain, pin./in.	100	177	2690	38	00 90 00 507	5 6450	7990	9710	12070	14440
48.	Strain, win./in.	118	243	390	5	4 72	5 909	1104	1312	1581	1840
	Strees, pel	1190	2430	3900	54	40 726	ນ 9090 ເ 170	11040	13120	15810	18400
h-16	Hement, 10/18.	0.01	0.03	0.05	0.0	27 0.0	0.11	0.12	0.14	0.15	0.16
	· · · · · · ·			-		0 00	. <u>.</u>	0.027	0.033	0.037	0.044
DC). 1159	Deflection, in.	0.002	0.003	0.000	0.0	0.00	5 0.007	0.010	0.011	0.013	0.015
1-1a:3-34	Ave thrust, 1b/in.	42	76	109	บ	1 17	3 209	249	266	336	383
2-2414-44	Avg thrust, lb/in.	2.00 2.00	39 2.00	1.79	1.	57 II 52 I.4	1.40	1.35	1.27	1.20	1.14
	•										
			Test C.	- <u>) (Z -</u>	7/8 in.)						
1	Strein, pis./in.	243	503	694	9	02 105	B 1180	1386	1562	1701	1857
_	Strees, pel	2630	5030	6940	90	20 1058 25 118	0 11800 1 1447	13880	15620	17010	2604
<b>JA</b>	Stress, pls./18.	1960	4130	7080	9ê	50 1181	0 14570	16730	19150	22090	24040
1-18	Theves, 1b/in.	48	101	154	2	01 24	6 290	337	382	430	1469 0.29
	Humant, in16/12.	-0.02	-0.04	0.04	0.	u <u>t</u> 0.0	2 4.11	0.44	V. 1-	0.20	
3	Strein, sin./in.	863	1.654	2257	-			••		**	
-	Street, pei	8630	16540	-1270	-16	-198	7 -2317	••			
-	Stress, pel	-4450	-8910	-12700	-164	40 -19 <b>8</b> 7	-23170	••		**	••
3-34	Thrust, 1b/1s.	-0 53	-1.03	109	-	• ••	••		••	••	
					-					•	
2	Strein, Min./ib.	223	416	616	5 80	20 104 00 1014	6 1298 0 12980	15820	1039	••	**
	Strein, pin./in.	88.30 57	188	352	5	24 73	3 969	1237	1479	1771	2034
	Struce, sel	570	1860	35.**	59	0 733	0 9690	12370	14790	17710	20340
2-86	Threat, 1h/15.	-0.07	-0.09	+0.11	-0.	18 -0.1	3 -0.13	-0.14	-0.15	••	••
								1400	1993	10006	991 7
•	Strein, Min./in.	839	433	6110	80	10 1016	0 12370	16930	17230	19760	22170
- <b>ha</b>	Strain, Min./in.	4		226	3	77 9	1 7%	204	1190	1437	1667
<b>b. k.</b>	Stress, pai	<b>10</b>	960	2960	37	70 <b>76</b> 1 10 17	210	9040 272	750	375	10010
	Humani, in14/18.	-0.09	-0,14	-0.16	-0.	17 -0.1	8 -0.19	-0.81	-0.21	0.92	-0.22
	Baddana Anna An	-0 004	-0.011	-0 M4		17	9 .0.017	+0.014	.0.011	-0.010	-0.006
	Deflection, 18.	0.003	0.003	0.003	0,0	oh 0.00	6 0.008	0.010	0.012	0.014	0.017
1-1413-30	Ang thrust, 11/13.	<u>17</u>	9	m		a		80) 80)	16 16 1		
2-30/b-bb	any thrust, 11/15.	1.2	1.48	1.32 1.32		. <u>77</u> 10	· · · ·	<b>E 74</b>	**	**	**
	2 <b>.</b>									•	

				Table :	5.7					
raia,	Stress,	Thrust,	and	Noment;	Tests	¢-6,	C-7,	c-8,	c-9,	C-10

tt.

					144	104	0						1.22	1.67	
		_2			100	Test Co	<u></u> 6 (2 x 0	<u>172</u>	200	<u> </u>		_50_			
1	Strain, uin./is.	-508	-836	-312	555	1443	2033	2459							
18	Strein, win./in.	-5080	-6360 876	-3120	5550	1010	20330	332							
1-1 <b>a</b>	Stress, psi Thrust, 1b/in.	7090 22	8760 4	13120	793	270	7100 303	3520							
3	Strain, win./in.	9	74	259	435	602	741	615							
34	Stress, psi Strain, µin./in.	90 143	740 538	2590 523	4350 718	6090 887	7410 9 <b>8</b> 0	<b>815</b> 0 1047							
3-34	Stress, psi Thrust, 1b/in.	1430	3380	5230 86	7180 127	164	9800 189	10170							
2	Strain, µin./in.	297	0.11 451	613	809	943	1184	1249							
28	Stress, psi Strein, µin./in.	29 <b>7</b> 0 1 <b>27</b>	4510	6130 396	8090	9 <b>43</b> 0 673	118140 752	12490 873							
2-2	Stress, psi Thrust, 1b/in.	-270 -47	2640 79	3960	5150 146	6730 178	7920 217	8730 233							
4	Noment, in-15/18. Strain, win./is.	-0.97 208	-0.08 333	-0.09 448	-0.12	-0.11	-0.10 761	-0.15							
48	Stress, psi Strein, min./in.	20 <b>8</b> . 136	3330 293	430	5670 518	6720 660	7610 707	8380 850							
ليسليع	Stress, pai Thrust, 1b/in.	1360 38	.3 <b>99</b> 0 69	4300	5180 119	6600 147	172	8500							
1-1413-34	Moment, in1b/in. Avg thrust, 1b/in.	-0.03 20	-0.02 25	-0.01 9 <b>6</b>	-0.02 155	0.00 217	0.00 246	0.00 255							
2-2414-44	Avg thrust, 1b/in.	43	74 0.34	104 0.94	133	163	195	210 1.21							
	•					Test C-	7 (2 - 3)	16 in.)							
1	Strain, win./in.		-2180		1037		2202		3240	3549 35490	3767 37554	420A 38935			
la	Strein, uin./in. Streen, pui		738		237 2370		-223 -2230		-251	167 1670	530 5300	599 5990			
1-18	Thrust, 1b/in. Moment, in1b/in.		57 0.39		140		218 -0.98	•	329	409 -1.36	¥73 •1.31	524 -1.41			
3	Strain, win./im.		809		1518		2090		2520 25300		3082	•••			
3 <b>n</b>	Strain, win./in.		-419		-723		-1113		-1417	-1633	-1735	-1908			
3-34	Thrust, lb/in. Moment, in,-lb/in.		43 -0.50		-0.90		107		-1.59	134	142	44 47			· · · · ·
2	Strain, Min./is.		303		ideo Altra		620		866	1185	137	2626			
28	Strein, pin./in.		200		10		790		963	1337	1547	-1814			
2-24	Thrust, 15/1n.		<b>6</b>		108		14 <b>6</b> 0.04		204	277	321 2.4f	376			
<b>k</b> <sup>2</sup>	Strain, uin./in.		306		559		1		1073	1449				in dia	
4a .	Strain, pis./is.				393		736		106	1.06	157	1873			
4 4.4	Thrust, 11/18.		- 65 -0.01		116 10.0-	•	346		215	14	399	4.04		•	
1-1413-3a	Ave thrust, 11/1s.		50		114		163		20	-		· • •			
8-2413-44	Ang thruch, 13/18.		0.77		109		1.0h	÷	1.00	G.92	<b>9</b> 40	•••			s. 191
					att an ar At	Den C	<u>4 (1 = 5</u>	06.16.)		1					
3	Strain, pin./in. Strees, pei		ц. Д		97) 97%		1943 19430		- Mars		21090 21090				107
1.	Strone, pai		3700		7		610	*		Linke	17100	20396	-		
1-18	Remains, inib/in.		0.4		-0.15		-9.74			-9.2	-0,16		0.95	9.85	4.4
3	Strain, uin./in. Strain, pil		57		199						10050	1134			Halling .
30	Strein, sin./in. Street, pri	ст. 	110 110		636		814 8144			NUMPS	14.36	1714	15540		17000
<b>j-ja</b>	Barort, 15/1a. Hamout, 1815/1a.		0.30		0.11		13A (2.09)		3.01	0.54	0 10	6.18	9 1 <del>3</del>	9 LJ	0.14
2	Strein, min./in. Strein, pil		3480		942 940		810 8100	a a si si ji	1091 10910			1751		22	Pho:
24	Mrain, pla:/is. Strand, pa.		LU LE		991 991				1093		1005		1113 11130		
2-84	Thrust, 15/16. Wommit, 1810/18.		70 0.90		10) 9.00	tan T	0.01		2 <b>36</b> 0.92	<b>30</b>	-0.08	ः <b></b>	-4.35	1988 1980	793 -0,34
•	Miraia, ula./la.		100				(M)10		LITS.			2107			
<b>M</b>	Strain, min./im.		UC HOD		803 8030	en en Second	1111		1540	17740	2141 21415	2346			
anta	Turnet, 10/18. Mennet, is10/18.		91. 0.02		145 9.04		0.01		0.00	578 4.05	9. <b>ch</b>	54.T	3.05	9. <b>9</b>	175
ية الإخر دواجة	Are therest, 12/14.		22		142. 1842		177			-	19-		1	z	2
4-4 <b>4</b> 79-98	my unter, My18.								0.05	<u>مک</u> ه				0.17	

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## Table 5.7 (Concluded)

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	a and a subscription of the statement	متقدي والتجرب	ميسينية مستغلمين و							<u> </u>		
		~	4		19.10.1							
1	Sibala, pip./in. Menue. pai	80	730 730	).266 12683		1603	1854	2019	2187 21890	2382 2380	2549	2675
<b>16</b> ·	Surain, wis./is.	565	863	1046		1206	1527	1992	2314	2679	بالجوري أ	3184
2-20	Warnet, 15/1a.	71	0630 176	30%60		12060	19210	19980	23160	26790	29180 601	31660
	House, in -th/in.	0.19	0.05	-0.09		-0.13	-0.13	-0.01	0.05	0.12	0.15	0.21
3	Strein, pig./in.	384	162	780		983	1127	272	1300	1532	1691	1807
1.	Werene, poi Morein, win./in.	25		7000		9836	14270	12720	13000	15320	10910	18070
	Bares, pti	2940	100	75-0		8360	101 30	.2190	14100	16010	17180	18000
3-34	Hennik, in10/in.	95 0.05	0.01	167 +2.02		40.0⊷	235 	274 -0.02	308 0.01	345 0.03	0.01	0.05
2	tereta, pis./1a.	-01	780	iola		1 160	1753	~ <b>**</b>	2509	294.	1942	1759
•	Street, pel	1010	7800	10820		1,96030	17080	20920	75090	29410	33420	37320
<b>-</b>	Person, per	473 1730	710	10010 10010			16370	2066	29120	2000	11683	35640
2- <b>31</b>	Barant, 15/1m.	20	165			Å	12	*57	541	632	716	805
	WEIMING, 18SE/18-	0.03	-0.03	- <b>4</b>		*12°, 14°	-14 <b>CS</b>	-0.01	=C.QA	•0.05	-0.07	- 49 A <b>R</b>
*	Strain, sta./is.	358	667 663	<b>3</b> 45		115i 118i	 t≿:∞.te:	1013 	2116	ALC:		2013) 2013
₩ <b>A</b>	Strato, sis /is	816		نوع		1242	1.54	1619	1579	238	240	2.00
bulks.	Birnis, pai	7160	54395 14	83.00		(412) - 412)	27,30	162.90	10790	21850	SAL SAL	25500
	Nament, in1h/in.	-0.07	- <b>9</b> £	-17.74		-0.93	-3.5	-0.09	-0.10	~`` <b>0</b> 9	-0.11	),33
1-1a13-3-	ang threat, ib/in.	58	2942			-61	<b>30</b> 4	350	·	431	-	585
1-10-10-10	ang Usrun, Style. 4	9.73				269) Q. 30	0.91	0.85	0.82	949 © <b>79</b>	039 0.16	<b>1</b> 01 0,75
				· · · ·		••						
				<b>NES Color (2 + 1</b>	11 H.)					ter i s		
¥	Strein, sin./in.	<u>i</u>	\$30	944		1154		1560	1757	1969	2181	2007
14	Renia, pla /1a.	505	25	1112		1992	13760	19800	2319	2563	21010 2777	
1-10	Strong, pet	5050	<b>1</b> 40	13140		15550	1000	active.	51190	25630	TIN	10810
5×46	Renaut, 3a12/1a.		9.1 <b>7</b>	9.3 <u>1</u>		1	0.18	8-3 <b>9</b>	3.0	19. <b>P</b>	., <b>≻</b> ⊋ ⊈, <b>₩</b>	¢. <b>3</b> 0
3	Brola, sia-/ia-	194		<del>34</del> 3		10	840	***	1.144	1277	1366	1540
<b>1</b>	Stress, pai	660	31300 Alto	54400		- <b>200</b> 0	1216	2470	1140	17140	13000	- 151 <b>8</b> 0
~	Mrten, pti		\$1.3			202 🕾	-	240	76110	7		6 M
3-36	Burnet, Myla	a 10	108 	157		一 彩 .		<b>215</b>	303	<b>31</b>		
			-									*
	Street, all /35	in the second	7140			1144 1124 :	1791	ACT	2,997	1715 1915-0	1000	
🖍 – Star	Montes, uts /in	<b>842</b>	377	<b>2</b> ]¥		1141				25 A	10	124.7
	Revel, pil		<b>3789</b> 936	2: <b>.</b>			157.4	410	9 <b>8.490</b> 523		677% 6%	19475
	Annal, in the		*	16, 58		-4 A	18 <b>M</b>		-2. <b>X</b>	-2 🕱	-10 🏚	-e. 🍅
•	Mirsia, sta /ia	A4)	20	1094		03	Tat		<b>8</b> .855	-	200	1.798
<b>4</b> 6 - 1997	Mirala, sla./15	5798() 3 <b>8</b> 89	1399					1. A				
	Revent, 201)	<b>4</b> 1	57.90°	Bas		10.00			1.41	<b>RTY</b>	-	
	Names, 10 - 10/10.	· · · · · · · · · · · · · · · · · · ·	14 - 14 - 14 - 14 - 14 - 14 - 14 - 14 -			1. 18			*R:	19 AN	-3.1T	\$ <b>™</b> . ₩.¥
1-1013-30	Ang themat, th/ta	<b>5</b>	(2)	200				1	178	113	-	-
-	Ang tigrant, 13/10.			C1+		<b>8</b> 5	2	<b>A</b> 9. <b>4</b>	-			<b>T</b> .)
	3	* **		¥-19		1.4	20 <b></b> .	U 78	.e. 24	¥: <b>7</b> 3	49 <b>F</b>	6. 189

			Tel	ble '	5.8				
Straia.	Stress,	Thrust,	Honest,	<b>1</b>	Deflection;	lests	8-1,	8-2,	§- }

	Hing automat	50	100	150	115	<u>200</u>	205	<u>11</u>	X	350	202	<b></b>
				1	Test 8-3 (8	+ C 1n.)						
1	Strain, uin./in. Stress. mi	60k 14903	1361 6308	2995 7927	4193 8111		11000					
2	Strain, uin./in.	- 397	-620	-9-6	-1179							
1-2	Thrust, 10/in.	62	140	2.	266							
4	Strain, Mis./15.	סر.ر مؤېر	-169	•.6	-298							
2	Stress, pol	-960	-1690	-1980	-2660							
ч 3 <b>ж</b>	Stress, pol	uoo	5334	64.87	••							
	Nummt, in16/16.	-1.42	-2.64	-2.79	••		**					
5	Birais, wis./is.	<del>ية.</del> 20-	70 200	239). 2390	320		56C					
£	Strain, uin./in.	264	346	445	522		691					
5-6	Thrust, 1h/in.		135	222	267		322	1. A.				
. ·	Mumant, Lt16/14.	+1,09 51	49.97 130	-0.73 201	-0.56 175		-C.12					
•	Stream, pai	\$10	1390	2210	3700		akaç Xes					
· .	Stress, pol	910	1560	1090	2280		- 2580					
	- Monent, 18/15. - Monent, 1812/14.	-0.16	98- ⊶⊂.06	3.04 1	4.50		22# 9.67					
e	Strum, ais./is.	2000 1000	*(1)	-91	600		58-0					
0	Strain, uis./is.	-206	-220	-202	-189		-176			. ·		
w10	Thrust, 16/14-	-2060	-2800	-20242 93	-1.090 127		-17 <b>60</b> 183					
	Homent, in16/10.	1.71	2.19	2.42	2.58		2.60					
	Strees, Sti	1300	2710	5779 59796	à <b>6</b> 75		100 P					
2	Pireis, uis./is. Piress, pil	₩. ₩0	50. 5 <b>40</b>	59 596	64.		68 680					
1-14 1-14	Thrust, 16/1s. Manast, 1s18/1s	- 5 <b>5</b>	1.13 (1.13	1.20	4.53		205			· ·		
<b>≇</b>	Persis, all fis		-	172	172		76E -					
ò	Birals, sis file	- <b>1965</b> -	- CBD 	4510	*3 <b>2</b> 5		• <b>35</b> 77 92 -			-		
1-34	Birnet, 10/1e	4090	474) 146	11.15 11.15 11.15	3474	•	11 19			· · ·		
	Houses, is, -13/14	+1.74	•k. 19	-t. č.	-0.29		-0.14		1			
Č	- Perlin, win./ik: - Person, pai		- 2015	-1 <sup>6</sup> 40			296 2960					
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ar an an a	dadže 41mm, in. dag zverset	(?) (363) ≪₹	24 645		195		**					
-40.13×1×	Ang to wet	9-9-1 	(43) (11)	194 101	2.6 <sup>1</sup>							
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4		N# 1.	791			2204		344	4.998	<b>13</b>	-	ta en la com
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	Harris 30-+10-1	\$ ¥	3.34	a. 🗮		1. 12		1.4	<i>ç</i> ;≜≸	2.55	1.1	
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	Sirain alls faith	48. 35 at 1	4.77	44-35 2014				17 <b>2</b>	•••• •••		3798	
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•	States, all The	17	<b>347</b>		1.14	<u></u>		1256		- <b>2011</b>	127	
<b>*</b> .	Martin all 24	417 <b>907</b> 544	<b>1</b>			<i>(</i> ***		1	186.4	192.	2429	
7 <b>-4</b>	Thrus, Main.		133	514 1	1	174	1111	1977 1977		195	1 1 <b>1</b> 1 1	
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<b>.</b>		-		-			an an art	-	41%¢	1445.3	MAR 1	
P\$	Verse, st.736.	*2 <b>59</b>	-1446	-143					1		794 5995	
	Mariat, 35/30 Million, 3135/5-		. <b>15</b>	1.39 2.34		1907 11-191		2006 1.394	13 13 14 14 14 14 14 14 14 14 14 14 14 14 14		*)# *. <b>44</b>	
LE	Strein, sin./is	144		191			· .·	1.54	3.400	*** Z	-	
1 <b>4</b>	Status, pit Status ola /la	**** *				99423 ···	· .		176	GR.	199	
11-12	Street, Still								100 A	8993 1937		
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Table	5.8	(Concluded)

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	Gase	Heaturement	30	100	150	175 200	205 250	300	350	130	440
1)       Breas, int./in.       -36       1.00       3.00       1.00 <td></td> <td></td> <td></td> <td></td> <td>Test 1-2</td> <td>(z = 7/16 in.) (Contin</td> <td>ued)</td> <td></td> <td></td> <td></td> <td></td>					Test 1-2	(z = 7/16 in.) (Contin	ued)				
Li deres, jul, no. ger webe 71.6 102: 107. BORD Web 21.6 107. 107. BORD Web 21.6 107. 107. 107. 107. 107. 107. 107. 107.	13	Strain, uin./in. Strass. pai	-92 -920	-12 -120	161 1610	518 4678	131	3 2786 7409	433. 816-2	5668 8734	
13-14         Threat, 1/2, 1/2, 1/2         1/2 <th1 2<="" th=""> <th1 2<="" th="">         1/2</th1></th1>	14	Strain, uin /in-	327	480 4579	716 5.95	1091 (033	199 695	5 <b>3690</b> 5 7875	52(9 8538	6512 3082	
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	13-14	Thrust, 15/in. Nombut, in15/in.	76 -1.48	152 -1.72	254 -1.28	352 -0.51	42 -0.2	497	543 -0.13	\$79 -0.12	
te strate, je	15	-alo, win./iz.	34	64	165	434 134	122	2701	3821 7040	5385 2612	
1         Proves, pri/A         170         1.00	16	Strair, Mis./in.	173	302	590	1138	2310	4306	6163	7953	
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	15-16	Stress, pai Thrust, 15/in. Mawent, in15/in.	1730 67 -0.49	3020 121 -0.82	4467 234 -1.22	606). 348 ~0.61	714. 43: -0-3	6 -0.28	549 -0.35	9000 595 -0.33	
Date that, dir.         Const.         <	DC1	Deflection, in.	0.017	0.026	0.035	0.043	0.05	5 0.070	0.086	0.105	
5-b:13-14         Arg Linut.         75         193         262         304         196         700	1-2:9-10	Avg thrust	4.	90	157	231	31	381	431	484	
International statement internatis internat	5-6:13-14	Avg thrust	75 0.55	149 0.60	262 0.60	354 0.65	43 0.7	3 490 2 0.77	542 0.80	0.83	
1         Breaks, siz./s.         565         635         347         1277         1.656         1989         2530         2533         2850           2         Breaks, siz./s.         3650         -273         -266         -211         -360         579         564         7757         126           12         Breaks, siz./s.         -266         -211         -360         590         540 <td></td> <td></td> <td></td> <td></td> <td>Tos</td> <td>t B-1 (2 = 7/8 in.)</td> <td></td> <td></td> <td></td> <td></td> <td></td>					Tos	t B-1 (2 = 7/8 in.)					
2         ittuin, in., in., in., in., in., in., in., i	1	Strain, µiz./in.	365	635 635	917 5723	1237	163	6 1989 9 6040	2530 7262	3253 7656	3885 7972
1.2         bit frees, pit J, m.         c.200         c.200 <thc.200< th=""> <thc.200< th="">         c.200</thc.200<></thc.200<>	2	Strain, µin./in.	-226	-272	-266	-214	-9	6 29	324	753	1268
Lament, in., -1.b/in.         7.06         2.94         3.11         2.93         3.77         97         1.65         1.07         0.74         0.59           4         Stream, pin., -1.51         370         700         1480         3870         5906         6777         7482         7934         6773         947         652.4         7477         617.6         654.4         7472         810.6         6672         2925         644.9         500         510         571         457.5         2899         410.5         510         500         510         571         457.5         2890         422.2         -0.22         -0.22         -0.22         -0.22         -0.25         -0.25         5007         700.5         1300         2570         130.5         2570         703.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704.8         700.9         704	1-2	Stress, psi Thrust, 1b/in.	-2260	-2720 109	-2050	538 *51: J	-90 29	290	400	1442	470
3         3         3         100         140         300         200		Roment, in1b/in.	2.06	2.94	3.11	2.93	a.3 م	7 1.85 7 1801	1.07	0.74	0.59
4         6 train, uin./in.         151.         307         331.         977         1975.         2899         4179         5288         8000           3-4         2trems, pin.         130.         307         312.         217         327         4.1         465.         307.         30.0         517.         4.1         465.         307.         30.0         517.         4.1         465.         307.         30.0         517.         4.1         465.         307.         30.0         517.         4.1         4.1         465.         40.2         941.         517.         517.         50.0         70.4         746.         40.0         538.         1304.         2129.         32.56.         446.9         941.         75.0         105.1         2028.         3369.         466.6         60.7         75.0         75.0         75.0         105.1         2028.         3369.         466.6         60.0         91.3         1217.         152.2         288.0         368.0         376.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         776.0         777.0 </td <td>3</td> <td>Stress, pei</td> <td>3? 370</td> <td>700</td> <td>1460</td> <td>3870</td> <td>590</td> <td>6 6777</td> <td>7492</td> <td>7964</td> <td>8373</td>	3	Stress, pei	3? 370	700	1460	3870	590	6 6777	7492	7964	8373
3-4         Thrust, 1h/Ln.         61.         123         217         327         417         465         507         500         711           Barest, inly/n.         0.02	4	Strain, µin./in. Stress, psi	151 1510	307 307 C	531 4712	937 2775	195 65-	5 2099	4179 8105	5528 8672	680R 9205
	3-4	Thrust, 1b/in. Moment, in1b/in.	61 -0,40	123 -0.83	217 -1.23	<b>327</b> -0.56	41 -0.3	7 465 5 -0.23	507 -0.22	540 -0.25	571 -0.29
6         Stress, pin./tr.         271         1/30         2872         1/35         2008         1366         1002	5	Strain, uin./in.	-57	24. 71-0	190	538	130	4 2129	3236	4429	5451
Sires, pil         2730         1390         5107         603         6776         771k         8394         8943         9948           5-6         Mameri, inlb/in.         -1.16         -1.15         -0.49         -0.26         -0.24         -0.26         -0.28         -0.26         -0.28         -0.26         -0.28         -0.26         -0.28         -0.26         -0.26         -0.26         -0.28         -0.26         -0.28         -0.21         -0.48	6	Stress, pil Strain, µin./in.	-570 273	439	682	1051	202	8 3369	4866	6176	7392
Heneri, inlh/in1.16-1.15-0.49-0.26	5-6	Stress, pei Thrust, 1b/in.	2730 70	<sup>k</sup> 390 150	5107 257	6033 354	697 43	6 7714 0 480	8394 522	8943 558	9448 588
7       8 trees, psi       3040       456       609       943       1217       1522       2262       2891       3347         8       Stress, psi       3040       4516       1916       3877       847       1111       2202       3770       3871         8       Stress, psi       260       750       1780       3870       9487       603       7070       7567       1629       1910       120       120       120       120       120       120       1562       3949       518       6772       6685       918       1238       1689       6679       668       918       1238       1689       6679       668       918       1238       1689       6679       668       918       1238       1689       6679       668       918       1339       2059       2806       3565       5660       5660       5666       6699       471 <td>-</td> <td>Noment, in1b/in.</td> <td>-1.16</td> <td>-1.46</td> <td>-1.15</td> <td>-0.49</td> <td>-0,2</td> <td>6 -0.24</td> <td>-0.26</td> <td>-0.26</td> <td>-0.28</td>	-	Noment, in1b/in.	-1.16	-1.46	-1.15	-0.49	-0,2	6 -0.24	-0.26	-0.26	-0.28
8         Stress, pai.         260         750         1760         3870         5877         1111         2202         3370         3871           7-8         Turnst, lb/m.         1070         173         242         225         364         407         461         489         509           9         Stress, psi         1120         1120         1120         162         3490         666         1129         1629         1629         1313         1314         1414         1314         1314         1314         1314	7	Strein, µin./in. Stress, psi	304 3040	456 4516	609 4916	913 5712	121 616	7 1522 1 6472	2262 7121	2891 7468	3347 7703
7-3       Thrus, bl/m.       107       173       242       525       324.       107       143.       1463       1669       500         9       Stress, psi       11.20       1120       1120       1120       1126       349       686       1129       1629       3490       5118       6072       6582       6672         1       Stress, psi       11.20       1120       1120       1500       500       5055       5785       6183       6382       6772       6482       6772       649       4500       5055       5785       6183       6382       6799       9       313       3139       2059       2806       1493       1182       1199       -0.21       -0.04       0.07       0.03         11       Strain, µin./in.       72       187       2880       475       893       1339       2059       2806       356         12       Strain, µin./in.       52       124       195       357       665       1660       2360       1562         12       Strain, µin./in.       52       124       195       357       665       1047       1650       2390       315       158       11407       1648	8	Strain, µin./in.	26	75 750	178	387	82 548	7 <u>1111</u> 7 6063	2202 7075	3070 7565	3871 7966
9       Strain, uin./in.       112       112       162       349       666       1129       1629       1914         9       Stress, pi       1120       1120       1120       1620       3490       5118       6072       6582       6672         9       Stress, pi       11.02       1120       1120       1620       3490       5118       6072       6582       6672         9       P       Stress, pi       1560       1690       2940       4500       5055       5725       6183       6382       6779         9-10       Thrust, in/in.       720       -0.20       -0.64       -1.01       -0.49       -0.21       -0.04       0 07       0.03         11       Stress, pai       720       1870       2880       4555       5660       6266       6994       7481       7883         12       Stress, pai       520       1240       1950       3570       5115       5988       6644       7186       7588         11-12       Thrust, ib/in.       0.07       0.22       0.33       0.38       0.10       0.13       0.08       0.07       501         13       Stress, pi	7-9	Thrust, lb/in. Noment, inlb/in.	107	173	242	325 0.54	36	4 407 3 0.15	461 U.02	489	509 -0.09
It         Stress, pai         pai.         file         file <thfile< th=""> <thfile< th=""></thfile<></thfile<>	9	Strain, µin./in.	112	112	112	162	34	9 686	1129	1629	1914
Brees, pai         560         1690         2940         4500         5055         5725         6133         6522         6743           9-10         Thrust, 1b/in.         157         91         132         199         294         352         396         421         444           Moment, in1b/in.         0.20         -0.64         -1.01         -0.49         -0.21         -0.04         0 07         0.03           11         Stress, pai         720         1870         2886         475         893         1339         2059         2808         352           12         Stress, pai         720         1870         2886         475         665         1047         1650         2390         315           Stress, pai         520         1240         1950         3570         5115         5988         6644         7182         7580           11-12         Thrust, 1b/in.         0.07         0.22         0.33         0.38         0.19         0.10         0.13         0.08         0.02           13         Stress, pai         -33         49         262         687         2021         2781         4057         5464         6528	10	Strain, µin./in.	56	169	294	450	66	2 918	1238	1433	1842
Homent, inlb/in.       0.20       -0.20       -0.64       -1.01       -0.49       -0.21       -0.04       0 07       0.03         11       Streas, pai.       17.0       187       288       475       893       1339       2059       2808       3566         12       Streas, pai.       720       1870       2880       4565       5665       5666       62946       6994       742.7       7823         12       Streas, pai.       520       1240       1950       3570       5115       5988       6644       7182       7580         11-12       Thrut, lb/in.       40       101       157       2669       350       399       444       475       501         13       Streas, pai.       -33       49       262       687       2021       2781       4054       8648       9069         14       Streas, pai.       -330       490       2620       5121       6977       8054       8649       9069         18       Streas, pai.       2810       453       5202       1345       7206       7646       8054       9069         13       Streas, pai.       2806       690       1520	9-10	Stress, pei Thrust, 1b/in.	560 55	1690 91	2940 132	4500 199	29	5 5725	398 9183	6362 421	6799 կեր
11       Strein, µin./in.       72       187       288       475       693       1339       2059       2808       3968         12       Stress, psi       720       1870       2880       4565       5660       6286       6994       7421       7883         12       Stress, psi       520       1240       1990       3370       5115       5988       6644       7182       7583         11-12       Thrust, 1b/in.       40       10       157       269       350       399       444       475       560         13       Stress, psi       -333       49       262       687       2021       2781       4057       5464       6528         3tress, psi       -333       49       2620       5121       6972       7405       8054       8645       9069         14       Strein, µin./in.       2810       455       718       1201       2431       3674       5088       6501       7575         13-14       Thrust, 1b/in.       -1.11       -1.43       -0.86       -0.38       -0.08       -0.16       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15		Moment, in10/in.	0.20	-0.20	-0.64	-1.01	-0.4	9 -0.21	-0.04	0 07	0.03
12       Strain, µin./in.       52       124       195       357       685       1047       1690       2390       315         Stress, psi       520       1240       1950       3570       5115       5988       6644       7182       7588         11-12       Thrust, 1b/in.       40       101       157       269       350       399       444       475       501         Mement, inib/in.       0.07       0.22       0.33       0.38       0.19       0.10       0.13       0.08       0.02         13       Strain, µin./in.       -33       49       262       687       2021       2781       4057       5464       6528         Stress, psi       -330       490       2620       5121       65772       7405       8054       8649       9069         14       Stress, psi       2810       453       5202       1145       2706       7867       8487       9078       9524         13-14       Thrust, 1h/in.       -1.11       -1.43       -0.86       -0.38       -0.06       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15       -0.15	11	Strein, µin./in. Stress, psi	72 720	187 1870	288 2880	<b>475</b> <b>456</b> 5	566	0 6286	2059 6994	2808 7421	7823
11-12       Thrust, ib/in.       40       101       157       269       350       399       444       475       501         13       Btreast, mlb/in.       0.07       0.22       0.33       0.38       0.19       0.10       0.13       0.08       0.02         13       Btreast, min./in.       -33       49       262       687       2021       2781       4057       5464       6528         Btreast, pin./in.       -33       490       2620       5121       6572       7405       8054       8645       9069         14       Btreast, pin./in.       2810       455       718       1201       2431       3674       5088       6501       7575         313       5202       1345       7206       7867       8487       9078       9524         13-14       Thrust, in./in.       16       280       690       152       388       1053       1953       2964       4114       4972         15       Btreast, min./in.       280       690       152       388       1053       1953       2964       4144       4972         16       Btreast, pai       1560       3410       4222       6623 </td <td>12</td> <td>Strain, µin./in. Stress. pai</td> <td>52 520</td> <td>124 1240</td> <td>195 1950</td> <td>357 3570</td> <td>68 511</td> <td>5 104γ 5 5988</td> <td>1690 6644</td> <td>2390 7182</td> <td>31 .5 7588</td>	12	Strain, µin./in. Stress. pai	52 520	124 1240	195 1950	357 3570	68 511	5 104γ 5 5988	1690 6644	2390 7182	31 .5 7588
13       Strain, µin./in.       -33       49       262       687       2021       2781       4057       5464       6528         Btress, psi       -330       490       262       5121       6972       7405       8054       8645       9069         14       Stress, psi       281       455       718       1201       2431       3674       5068       6501       7575         31-14       Thrust, 1b/in.       81       164       281       372       461       497       538       576       605         15       Stress, psi       280       690       152       388       1053       1953       2954       414       497       538       576       605         15       Stress, psi       280       690       152       388       1053       1953       2954       414       497       538       576       605         16       Stress, psi       280       690       1520       3880       1953       1953       2954       414       4979       538       9300       15-15       6023       6997       7587       8273       8865       9340       15-16       Stress, psi       1560       341 <td>11-12</td> <td>Thrust, lb/in. Moment, in1b/in.</td> <td>40 0.07</td> <td>101 0.22</td> <td>157 0.33</td> <td>269 0.38</td> <td>39 0.3</td> <td>0 399 9 0.10</td> <td>444 0.13</td> <td>475 0.08</td> <td>501 0.08</td>	11-12	Thrust, lb/in. Moment, in1b/in.	40 0.07	101 0.22	157 0.33	269 0.38	39 0.3	0 399 9 0.10	444 0.13	475 0.08	501 0.08
14       Btrain, µin./n.       291       455       718       1201       2431       3674       5088       6501       7575         Btress, psi       2810       4513       5202       1345       7206       7867       8487       9078       9524         13-14       Thrust, 1b/in.       81       164       281       372       441       497       538       576       605         15       Btrain, µin./in.       28       69       152       388       1053       1953       2964       4141       4972         16       Btrain, µin./in.       156       69       152       388       1053       1953       2964       4141       4972         16       Btrain, µin./in.       156       341       611       1081       2048       314       4579       5987       7133         15-16       Thrust, 1b/in.       60       133       234       339       423       477       514       551       576         15-16       Thrust, 1b/in.       60       133       234       339       423       477       514       551       576         15-16       Thrust, 1b/in.       0.044       0.027 <td< td=""><td>13</td><td>Strain, µin./in. Stress. pai</td><td>-33 -130</td><td>49 290</td><td>262 2620</td><td>687 5121</td><td>202 697</td><td>1 2781 2 7405</td><td>4057 8054</td><td>5464 8645</td><td>6528 9089</td></td<>	13	Strain, µin./in. Stress. pai	-33 -130	49 290	262 2620	687 5121	202 697	1 2781 2 7405	4057 8054	5464 8645	6528 9089
13-14       Durves, pi:       2010       712       710       710       700       713       713       713       713       713       710       710       710       710       710       710       710       713       710       710       710       710       710       710       710       710       710       710       710       710       710       710	14	Strain, µin./in.	281	455	718	1201	243	1 3674 6 7867	5088	6501	7575
15         Strain, μin./in.         28         69         152         388         1053         1953         2964         4141         4972           Stress, psi         280         690         1520         3880         5994         6912         7509         8089         8438           16         Stress, psi         1560         341         1081         2048         3114         4579         5987         7133           Stress, psi         1560         3410         4922         6023         6987         7587         8273         8865         9340           15-16         Thrust, 1b/in.         60         133         234         339         423         477         514         551         576           Moment, in1b/in.         -0.45         -0.96         -1.28         -0.66         -0.36         -0.23         -0.27         -0.32           DC1         Deflection, in.         0.014         0.027         0.033         0.042         0.050         0.070         0.079           DC2         Deflection, in.         0.013         0.017         0.020         0.025         0.046         0.060         0.025         0.050         0.070         0.079 <tr< td=""><td>13-14</td><td>Thrust, lb/in. Moment, inlb/in.</td><td>81 -1.11</td><td>164 -1.43</td><td>281</td><td>372 -0.38</td><td>46 -0.0</td><td>1 497 18 -0.16</td><td>538 -0.15</td><td>576 -0.15</td><td>605 -0.15</td></tr<>	13-14	Thrust, lb/in. Moment, inlb/in.	81 -1.11	164 -1.43	281	372 -0.38	46 -0.0	1 497 18 -0.16	538 -0.15	576 -0.15	605 -0.15
16       Strain, uin./in.       156       341       611       1081       2048       3114       4573       5967       7133         15-16       Stress, psi       1560       3410       4922       6023       6987       7587       8273       8865       9340         15-16       Thrust, lb/in.       60       133       234       339       423       477       514       551       578         Moment, in.       10.45       -0.96       -1.28       -0.66       -0.36       -0.23       -0.27       -0.27       -0.32         DC1       Deflection, in.       0.014       0.021       0.027       0.033       0.042       0.050       0.060       0.070       0.079         DC2       Deflection, in.       0.013       0.017       0.020       0.023       0.042       0.050       0.060       0.070       0.079         1-2:9-10       Avg thrust       50       100       150       214       293       346       399       431       457         5-6:13-14       Avg thrust       76       157       269       363       446       489       70       567       597         q       0.666       0.64	15	Strain, µin./in.	28 280	69 690	152	388	105	3 1953 4 6012	2364 7400	4141 8080	4972 RL 78
Stress, psi         1500         3410         4922         6023         6967         7567         8273         8865         9340           15-16         Thrust, 1b/in.         60         133         234         339         423         477         514         551         578           Moment, in1b/in.         -0.45         -0.96         -1.28         -0.66         -0.36         -0.23         -0.27         -0.27         -0.32           DC1         Deflection, in.         0.014         0.021         0.027         0.033         0.042         0.050         0.060         0.070         0.079           DC2         Deflection, in.         0.013         0.017         0.020         0.023         0.025         0.46         0.029         0.030           DC2         Deflection, in.         0.013         0.017         0.020         0.023         0.025         0.46         0.399         431         457           1-2:9-10         Avg thrust         50         1000         150         214         293         346         399         431         457           5-6:13-14         Avg thrust         76         157         269         363         446         489	16	Strain, uin./in.	156	341	611	1081	20	8 3114	4579	5987	7133
Moment, inlb/in.         -0.45         -0.96         -1.28         -0.66         -0.36         -0.23         -0.27         -0.27         -0.32           DC1         Deflection, in.         0.014         0.021         0.027         0.033         0.042         0.050         0.070         0.079         0.079           DC2         Deflection, in.         0.013         0.017         0.020         0.023         0.025         0.46         0.022         0.039         0.030           DC2         Deflection, in.         0.013         0.017         0.020         0.023         0.025         0.46         0.029         0.030           DC3         Avg thrust         50         100         150         214         293         346         399         431         457           5-6:13-14         Avg thrust         76         157         269         363         446         489         70         567         597           q         0.66         0.64         0.56         0.59         0.66         0.71         0.75         0.76         0.77	15-16	Stress, psi Thrust, lb/in.	1560	3410 133	4922	6023 339	698 42	n 7587 13 472	8273 514	551	9340 578
DC1         Deflection, in.         0.014         0.021         0.027         0.033         0.042         0.050         0.071         0.071         0.071         0.073         0.023         0.025         0.044         0.033         0.025         0.031         0.075         0.071         0.075         0.077         0.077         0.071         0.075         0.077         0.077		Moment, in1b/in.	-0.45	-0.96	-1.28	-0.66	-0.3	6 -0.23	-0.27	-0.27	-0.32
1-2:9-10         Avg thrust         50         100         150         214         293         346         399         431         457           5-6:13-14         Avg thrust         76         157         269         363         446         489         70         567         597           q         0.66         0.64         0.56         0.59         0.666         0.71         0.75         0.76         0.77	DC5 DC1	Deflection, in. Deflection, in.	0.014	0.021 0.017	0.027	0.033 0.023	0.04	e 0.050 5 0.(⊖6	0.02	0.070	0.079
q 0.66 0.64 0.56 0.59 0.66 0.71 0.75 0.76 0.77	1-2:9-10 5-6:13-14	Avg thrust Avg thrust	50 76	100 157	150 269	214 363	29	13 346 16 480	399 ""0	431 567	457 597
	, <u>.</u>	q	0.66	0.64	0.56	0.59	0.6	6 0.71	0.75	0.76	0.77

5.6

and the second

Gege	Messurement	Test E-6 Z = 0 in. P = 254 psi so = 254	Test E-5 Z = 7/16 in. P = 262 psi sc	Test E-4 Z = 7/8 in. P = 264 psi so
1	Strain, µin./in.	5841	7 <b>899</b>	5698
	Stress, psi	8803	9658	8743
2	Strain, µin./in.	8637	4881	3849
	Stress, psi	9933	8400	7955
1-2	Thrust, lb/in.	610	587	543
	Moment, inlb/in.	-0.40	C.44	0.27
3	Strain, µin./in.	3207	3949	3326
	Stress, psi	7634	8004	7693
4	Strain, µin./in.	5131	6189	5344
	Stress, psi	8505	8948	8594
3-4	Thrust, lb/in.	526	551	531
	Moment. inlb/in.	-0.30	-0.33	-9.31
5	Strain, µin./in.	4197	5582	4802
	Stress. psi	8113	8694	8367
6	Strain, µin./in.	4654	4946	5392
	Stress, psi	8305	8427	8615
5-6	Thrust, lb/in.	534	556	552
	Moment, in_lb/in.	-0,07	0.09	-0.09
7	Strain, µin./in.	5878 8819	7562 9518	6045 8889
8	Strain, µin./in.	3944 8002	59 <b>04</b> 8830	4565 8267
7-8	Thrust, lb/in.	547	596 0.24	558 0.22
9	Strain, µin./in.	5098 8401	6552	5300 8576
10	Strain, µin./in.	3622 7841	4991 8446	3653 7856
9-10	Thrust, lb/in.	532	570 0.23	535 0.25
u	Strain, µin./in.	4415	6537	4503
	Stress, psi	8204	9093	8241
12	Strain, µin./in.	5347	5148	8404
	Stress, psi	8596	8512	1890
11-12	Thrust, lb/in.	546	572	541
	Moment, inlb/in.	-0.14	0.20	-0.06
13	Strain, µin./in.	3233	5350	3757
	Stress. psi	7647	8597	7909
14	Strain, µin./in.	3949	4969	4297
	Stress. DSi	8004	8437	8155
13-14	Thrust, lb/in.	509	554	522
	Moment, inlb/in.	-0.13	0.06	-0.09
15	Strain, µin./in.	2651	3581	3630
	Stress, psi	7331	7821	7845
16	Strain, µin./in.	4898	6172	5499
	Stress. psi	8407	8941	8 <b>66</b> 0
15-16	Thrust, lb/in.	<b>51</b> 4 -0.38	546 -0.39	537 -0.28
1-2:9-10 5-6:13-14	Avg thrust Avg thrust	571 521	579 555 1.64	539 537 1.00

Table 5.9

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Table 5:10											
train.	Stress.	Thrust.	Moment,	and	Deflection;	Tests	D-1,	<b>D-2</b> ,	D-3,	p.4,	D-5

Gary	Measurement	25	50	75	90 95	pressure, p 100	130	150	175	180	190
				Te	$\frac{1}{2} = 0 \text{ in.}$					<u>مستختفي</u>	
L	Strain, uin./in.	293	684	2881	16194						
_	Stress, pai	2930	4249	62 <u>5</u> 5							
	Strain, uin./in. Stress. psi	-1640	-3677	-1693	-1036						
L-2	Thrust, 15/in.	35	58	179							
	Moment, in10/1h.	165	3.23	*•23	**						
3	Strain, µih./in. Stress, psi	->7 -570	120	270	-1/10 -5512						
•	Strain, µin./in.	195	224	402	7109						
1_ii	Stress, psi Thruct, lb/in.	45	2240 77	3703 137	278						
	Moment, in1b/in.	-0.89	-0.75	-1.25	-4.61						
5	Strain, µin./in.	190	-59	-542	-398						
5	Stress, psi Strain. uin./in.	-54	-7990	-3957 1303	-37/1 2142						
	Stress, psi	-540	2650	5161	5845						
<b>-</b> ••	Moment, in1b/in.	0.86	-1.14	-3.61	-2.89						
i	Strain, win./in.	88	-32	-12	268						
•	Stress, psi	380	-320	-120	2680						
0	Strein, µin./in. Stress, psi	-39	1120	2360	810						
7-5	Thrust, 1b/in.	16	26	73	113						
	Moment, inib/in.	0.47	•0.71 	-0.07	2000						
,	stress, psi	-1360	3403	5424	6380						
10	Strain, µin./in.	163	-222	-1073	-1718						
-10	Stress, psi Thrust, lb/in.	1630 9	-2220	-4963 67	-5517 108						
,	Moment, inlb/in.	-1.05	1.98	4.60	5.23						
12.	Strain, µin./in.	60	-6	-66	.90						
12	stress, psi Strain, µin./in.	-11	-60 81	-000	188						
	Stress, psi	-110	810	2370	1860						
11-15	Thrust, 1b/in. Moment. in1b/in.	16 0.25	-(0.31	>6 -1.07	-0.35						
13	Strain, uin./in.	525	389	504	642						
-	Stress, psi	3922	3544	2040	4162						
14	Strain, µin./in. Stress. Dai										
13-14	Thrust, 1b/in.										
	Moment, in1b/in.	at 0		1.17							
15	Strain, µin./in. Stress. psi	-348 -3424	-321 -3210	-417 -3627	- 3980 -6864						
16	Strain, µin./in.	799	1065	1725	9744						
15-16	stress, psi Thrust, 1b/in.	4434	4956	2525 191	220						
	Moment, in10/in.	-2.97	-2.38	-3.08	-6.43						
DC1	Deflection, in.	0	0.018	0.078	0.165						
1-2:9-10	Avg thrust	-0.002	49	123	0.010						
5-6:13-14	Avg thrust	0.50	0.73	0.04							
		0.30	V+(3	v.yu							
	Strate into the	••	1.74	1695	$z = y - \frac{1}{2} (z = \frac{7}{10} \ln \frac{1}{2})$	4.75.2	15400				
1	strain, µin./in. Stress. psi	-11 -110	476 3799	5403		6888	10634				
2	Strain, µin./in.	195	-251	-832		-1530	-9231				
1-2	Stress, psi Thrust, 1b/in.	1950	-2510	-4224 102		-2326 182	-09/- 140				
	Moment, in1b/in.	-0.73	2.39	4.22		4.84	8.54				
3	Strain, µin./in.	159	235	320		353	-738				
l,	Stress, psi Strain, µin./in.	1590 : -26	2350	3200 82		3430 256	4665				
- 1	Stress, pai	-260	560	820		2560	7156				
3-4	Thrust, 1b/in. Moment. in1b/in.	43 0.65	95 0.63	0.84		0.33	-3.30				
5	Strain, µin./in.	24 1:	-78	-476		-740	-266				
	Stress, pei	2440	•780	-3799		-4364	-2650				
0	Strain, µin./in. Stress, pai	-147 -1470	350 34457	5124		5781	6606				
5-6	Thrust, 1b/in.	32	90	134		158	261				
_	Noment, in15/in.	1.30	-1.92	-3.42		-3.%	•6.37				
7	strain, µin./in. Stress, psi		••					•			
8	Strain, µin./in.	341	407	445		428	422				
7-8	Stress, p#1 Thrust, 1b/in.	5403	22241	3700		3079	 				
	Moment, in1b/in.	**	••			•=	••				
9	Strain, uin./in.	74	285	618		1014	1490				
10	Stress, pai Strain. uin./in.	42	-114	-350		-506	-1,74				
	Stress, pai	420	-1140	- 3429		-3882	-4 022				
9-10	Thrust, 15/in. Moment, in15/in.	0.11	30 1.40	2.91		3.53	3.65				
11	Strain, µin./in.	13	153	392		717	1195				
	Stress, pai	130	1530	3553		4317	5068 -171				
19		103		تعنيه- بر			- 4 I <b>+</b>				
12	Stress, pei	1030	240	-1080		-1950	-1710				

(Continued)

Table 5.10 (Continued)

						Over	pressure, p	1				
Gege	Moastrement	25	50	75	90	St.	100	130	150	175	180	190
	<b>•••</b> • • •	Q**	261	Test D-2	(z = 7/16  tr)	n.) (Contin	<u>ued)</u> _891	- 397				
13	Stress, psi	870	-2610	-4156			-4635	-3538				
14	Strain, µin./in.	21 210	618 4113	1895 5670			3023 6372	4009 6873				
13-14	Tirust, 1b/in.	35	95	163			505	306				
	Moment, in1b/in.	0.23	-2.56	-3.71			-3.96	-2.51				
1)	Strain, pin./in. Stress, pai	70	1060	1730			2330	-4978				
16	Strain, µin./in.	147	199 1990	- 199 - 1990			387 3538	5444 7491				
15-16	Thrust, 16/1n:	50	99	121			198	278				
	Noment, in -1b/in.	-0.49	-0.33	-0.09			+0.46	-3.09				
DC1 DC2	Deflection, in. Deflection, in.	-0.009	0.017	0.094			0.072	0.088				
1-2:9-10	Avg thrust	44 34	62 03	85 140	:		139 180	140 282				
9-0:13+14	d VAR CUL 20	1.38	0.67	0.57			0.77	0.50				
				Tez	t D-3 (Z =	7/3 in.)						
1	Strain, uin./in.	32.9	2376	7863	13973							
	Stress, psi	3190 - 26ii	5986 -1290	8447 _2052	10502							
2	Stresz, psi	-2640	-5235	-6897	-9166							
1-2	Thrust, 1b/in. Moment, in1b/in.	18 2.05	96 4,95	159 6.57	115 8.66							
3	Strain. win./in.	-40	-222	-386	-1052		-9530					
- ).	Stress, psi		-2220	-3535	-4945		-9089 17656					
4	Stress, psi	1670	4062	5324	6395							
3-4	Thrust, 1b/in.	38 -0.76	102	178 -3.04	183 -4,30							
5	Strain. uin./in.	30	-438	-651	->33		166					
	Stress, psi	300	-3688	-4181	-3938		1660					
0	Strein, µin./in. Stress, psi	1180	4562	5284	5376		5077					
5-6	Thrust, 1b/in.	48	-3, 30	120 =3.86	153		265 -0,98					
7	Strain, uin /ir.	-136	⊌ر.ر_ يلبا	515	946		1211					
, ,	Stress, pai	-1360	440	3901	4789		5081					
8	Strain, µin./in. Stress. psi											
7 <b>-</b> 8	Thrust, 1b/in.											
٩	Strein uin An	271	527	703	802		841					
,	Stress, psi	2710	3926	4288	44.92		4572					
10	Strein, µin./in. Stress. psi	-261 -2610	-522 -3915	-657 -4193	-4193		-4255					
9-10	Thrust, 1b/in.	3	1	9	28		29 29					
11	Stamin win /in	-161	3.20 tou	871	1610		1504					
11	Stress, pei	-1610	1940	4634	5260		5333					
12	Strein, µin./in. Stress, poi	161 1610	-50	-440 -3717	-4053		-4068					
11-12	Thrust, 1b/in.	0	47	87	128		134					
12	Strein uin /in	-11	-230	- 225	-168		691					
-1	Stress, psi	940	-2300	-3250	-1680		4263					
14	Strain, µin./in. Stress, psi	-240	3805	4731	4971		4410					
13-14	Thrust, 1b/in.	23	-2.32	127	188		-0.05					
15	Strain, win./in.	-154	-308	-411	-1101		-7581					
•/	Stress, pai	-1540	-3080	-3609	-4987		-8339					
10	Stress, psi	2950	4296	5348	6502		10674					
15-16	Thrust, 1b/in.	46	98 -2.79	175	188 ના, સંઘ		195 •8.03					
<b>11C1</b>	Deflection. in.	0.013	0.058	0.142	0.171		0.186					
DC2	Deflection, in.	0.007	0.040	0.073	0.068		0.093					
1-2:9-10 5-6:13-14	Avg thrust	36	79	124	171		274					
	<b>q</b>	0.31	0.62	0.68	0.42							
				200	t D=4 (Z = )	<u>1-3/4 in.)</u>						
1	Strain, µin./in.		561 2006				2896 6298		5370 7459			9051 8905
2	Strain, µin./in.		- 348				-1201		-1584			-1966
1-2	Stress, psi Thrust, lb/in.		-3423 57				-2073		232			-212
•-•	Moment, inlb/in.		2.66				4.60		4.69			4.74
3	Strain, µiu./in.		143				211 2110		737 4358			756
4	strain, pin./in.		132				448		1453			5062
3-4	Stress, psi Thrust, lb/in		1320				203		320			403
<b>J</b> - ··	Moment, in1b/in.		0.04				-0.58		-0.3			-0.93
5	Otrain, uin./in.		-185 -1850				-1826 -5610		-2434 -6020			-2353 -9990
6	Strain, µin./in.		431				3553		6360			8480
5-6	Stress, psi Thrust, 16/18.		,9807 77				119		203			\$72
	Moment, in1b/in.		-2.08				-5.27		-5.51			-5.24
					10	( ممد						

## Table 5.10 (Concluded)

		······································	د	Overpressure, pa	1			
Gage	Measurement	<u>25 50</u>	<u>75 90</u>	<u>95 100</u>	130 150	175	180	190
7	Strain, win. /in.	-253	Test D-4 [2 = 1-1/4 ;	in.) (Continued) 250	896			150
8	Stress, psi Strein, uin./in.	-2530		2500	4542			533
7_8	Stress, psi	3441		1270	2360			4408
(-0	Moment, in10/in.	-2.13		0.43	0.63			0.30
9	Strain, µin./in. Stress. psi	318 3180		1346 5198	2366			3155 6439
10	Strain, µin./in.	-263 -2630		-665	-673 -4226			-459
9-10	Thrust, 1b/in.	18		106	192			265
ц	Strain, win./in.	478		1410	2397			3120
12	Stress, pai Strain, uin./in.	3805		5253 697	5998 -692			642) -468
11-12	Stress, pei Trust, 1b/in.	~2520 67		-4276	-4266			-3776
11-12	Moment, in1b/in.	2.40		3.97	3.70			2.8
13	Strain, µin./in. Stress, psi	-230 -2300		-1270 -5132	-1577 -5396			-1292 -5151
14	Strain, µin./in.	476 3799		3913 6826	6811 3042			9347
13-14	Thrust, 1b/in.	-2.32		201	279 -1 17			361
15	Strain, µin./in.	-175		223	811			1121
16	Stress, psi Strain, win./in.	-1750 Լկե		1230 360	4511 985			5004 3543
15-16	Stress, ps. Thrust, lb/in.	3706 83		3459 369	4869 305			6637
u) 12	Moment, inlb/in.	-2.06		<b>~0,4</b> 6	-0.13			-0.57
DC1 DC2	Deflection, in. Deflection, in.	0.026 0.023		c.080	0.093			0.100
1-2:9-10 5-6:13-14	Avg thrust Avg thrust	38 76		130 165	162 244			289 317
	e .	0.50		0.79	0.66			0.91
			Test D-5 (Z = 2	-5/8 in.)				
1	Strain, uin./in.	1591		7368	11022			
2	Strein, µin./in.	-871		8257 -4137	9587 -5661	<b>-990</b> 6		
1-2	Stress, psi Thrust, 10/in.	-4634		-6929	-7584 180	-9234		
2	Moment, in15/in.	4.29		6.63	7.28			
), ́	Stress, psi	-3567		-3913	-4313	-6204	-8105	
	Stress, psi	4842		5822	4472 7073	9350	20912	
3-4	Moment, inlb/in.	-3-18		-3.31	283 -3,28	296 -5.45		
5	Strain, µin./in.	-201		-331	-77	602	878	
6	Strain, µin./in.	361		-5510 998	1617	1669	1408	
5-6	Thrust, 1b/in.	52		4596	250	320	325	
7	Strain, uin./in.	-1.97 198		-2.92	-1.79 541	-0.46 660	-0.19 680	
R	Stress, psi Strein, uin, /in	1980		3930	3954	4200 104	4241	
7_8	Stress, psi	-1940		-2890	890 191	1940	2060	
/ <b>-</b> ()	Noment, in1b/in.	1.38		2.60	1.12	0.71	0.67	
9	Strain, µin./in. Stress, psi	468 3776		1404 5247	2635 6141	3160 6442	3160 6442	
10	Strain, µin./in. Stress, psi	- 381 - 3520		-1026	-1251 -5116	-1269 -5131	-1273	
9-10	Thrust, lb/in.	24		51	131	162	162	
11	Strain, µin./in.	-22		368	703	855	876	
12	Stress, pei Strain, µin./in.	-220 127		3482 8	4268 208	4601 389	4645 408	
11-12	Streas, psi Thrust. 1b/in	1270 34		90 12-	2080 233	3544, 267	3600	
	Moment, in16/in.	-0.52		1.25	0.67	0,36	0.35	
13	Strain, µin./in. Stress, pei	-127 -2170		-233 -2330	0	710 4303	1356 5206	
14	Strain, µin./in. Stress, psi	304 304:0		1000 4900	1588 5406	1795 5584	1726 5524	
13-14	Thrust, 1b/in. Moment, in1b/in.	58 -1.52		162 •2.51	261 -1.52	330 -0.41	349	
15	Strain, µin./in.	-228		-250	-323	-1688	-5159	
16	Streas, pel Strein, µip./in.	-2260 558		-2500 1344	-3230 3173	-5492 7639	-7368 14749	
15-16	Streas, pai Thrust, 1b/in.	<b>398</b> 9 93		5196 195	6448 285	8361 294	10742	
<b>1.4</b> .	Moment, in. =1b/in.	-2.40		•(2, 5) }	-2,42	-4.52	-6.95	
DC2	periection, in.	0.038 0.038		0,149 0,098	0,176 0,098	0.184	0.186 0.101	
1-219-10 5-6:13-14	Avg thrust	61 55		95 150	156 256	325	137	
	q	1.11		0.63	0.61			

Table	5.11	
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Strain, Stress, Thrust, and Moment; Tests D-6, D-7, D-8, D-9, D-10

Gage	Measurement	Test D-10 Z = 7/8 in. P <sub>so</sub> = 97*psi	Test D-8 Z = 7/8 in. P = 116*psi so	Test D-9 Z = 7/8 in. P = 148 psi so	Test D-6 Z = 1-3/4 in. P = 160# psi	Test D-7 Z = 1-3/4 in. P = 180 psi so
1	Strain, µin./in. Stress, psi		3361 6541	17296	13714 10421	19059
2	Strain, µin./in. Stress, psi		-891 -4675	-15970	-5512 -7520	-10134 -9312
1-2	Thrust, lb/in. Moment, inlb/in.		218 3.90		251 7.16	
3	Strain, µin./in. Stress, psi		343 3409	-14665 -1071.6	-5063 -7327	-7055 -8136
4	Strain, µin./in. Stress, psi		 	5836 7659	14 <b>38</b> 0 10 <b>628</b>	19105
3-4	Thrust, lb/in. Moment, inlb/in.		 	-7.30	283 -6.90	
5	Strain, µin./in. Stress, psi	••	1174 5-50	9087 8918	3 <b>88</b> 1 6809	5 <b>38</b> 4 7465
6	Strain, µin./in. Stress, psi		550 3973	17760 	<b>-66</b> 5 -4210	-686 -4253
5-6	Thrust, lb/in. Moment, inlb/in.		<b>298</b> 0.40		268 3.26	314 3.10
7	Strain, µin./in. Stress, psi	••	-53 -530	1468 5302	860 4612	-2511 -6067
8	Strain, µin./in. Stress, psi		1933 5702	-411 -3609	591 4057	6892 8073
7-8	Thrust, lb/in. Moment, inlb/in.		271 -1.68	170 3.14	262 0.20	216 -5.52
9	Strain, µin./in. Stress, psi	1551 5374	3195 6459		2865 6279	-1910 -5683
10	Strain, µin./in. Stress, p <b>s</b> i	-1306 -5163	-576 -4027		720 4 <b>323</b>	5327 7441
9-10	Thrust, lb/in. Moment, inlb/in.	29 4.83	250 3.18	••	358 0.62	204 -5.10
11	Strain, µin./in. Stress, psi	349 3426	-754 -4393		<b>88</b> 0 4 <b>653</b>	508 3886
12	Strain, µin./in. Stress, psi	0 0	3827 6782		3477 6603	295 2950
11-12	Thrust, lb/in. Moment, inlb/in.	113 1.23	255 -3.49		377 -0.64	231 0 <b>.28</b>
13	Strain, µin./in. Stress, psi	517 3905	<b>665</b> 4210		20 <b>7</b> 9 <b>58</b> 07	3998 6869
14	Strain, µin./in. Stress, psi	0	480 3811	••	1935 5704	206 2060
13-14	Thrust, lb/in. Moment, inlb/in.	154 1.47	<b>261</b> 0.14	••	374 0.04	362 1.18
15	Strain, µin./in. Stress, psi	-2171 - <b>586</b> 3	823 4535	••	-768 -4422	-11447 -9719
16	Strain, µin./in. Stress, psi	8019 8507	1105 4990	••	<b>597</b> 6 7720	22539
15-16	Thrust, 1b/in. Moment, in1b/in.	272 -5.06	312 -0.17	**	323 -3.21	
1-2:9-10 5-6:13-14	Avg thrust Avg thrust Q	 	234 2890 0.84		305 321 0.95	338

\* No failure.

.







 $M_c = 0.25 p_a R^2$ 



a. TALBOT (1908)

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b. CAIN (1929)







c. SPANGLER (1938)

d. WHITE (1960)

Fig. 2.1 Concepts of Load Distribution

102 Pso VERTICAL AXIS CROWN QUARTER POINTS 45 θ HORIZONTAL AXIS 90 SPRING LINE SPRINGING 270° MIDDLE SURFACE INTRADOS EXTRADOS INVERT 180" y Fig. 3.1 Cylindrical Shell and Ring Notation



Fig. 3.4 Nomuniform Lond Fig. 3.5 Elastic Supports Fig. 3.6 Elastic Medium





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a. Static Bonnet (2-ft Diameter, 500 psi)



b. Dynamic Bonnet



c. Spacer Ring and Posttest Diaphragm Configuration



d. Cylinder in Position

Fig. 4.2 University of Illinois Blast Load Generator





Fig. 4.4 WES Small Blast Load Generator (SBLG) Facility



Fig. 4.5 Illinois Instrum Intation Equipment

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Fig. 4.6 WES Large Instrumentation Room

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Fig. 4.7 WES Small Blast Load Generator (SBLG) Instrumentation













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Fig. 5.4 Stress, Thrust, Moment, and Deflection, Test A-3A (2 = 7/8 in.)

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DIAMETER CHANGE DECREASE PLOTTED POSITIVE INCREASE PLOTTED POSITIVE GAGE DC 1 DC 2 ¥ 010 DIAMETER CHANCE, IN 0.05 0 1.0 MOMENT, IN -LB/IN. ٥ ~1.0 -2.0L 500 f OFFSET YELD <u>Symeol.</u> O A D V CROSS-SECTION i-là 2-28 3-38 4-48 400 EMST. YELQ THRUST, LB/M. 300 -200 = DC I 100 £ 9714901. 0 4 4 5 6 7 7 7 10,000 GAGE ٠ 18 2 24 3 34 4 7.90 5000 STRESS, PS 210 NOO SUNFACE OVER



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450

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400





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Fig. 5.7 Stress, Thrust, and Moment, Test A-10 (Z = 0 in.)







Fig. 5.9 Stress, Thrust, and Moment, Test A-8 ( $Z = \frac{16}{16}$  in.)

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Fig. 5.11 Stress, Thrust, and Moment, Test A-6 (Z = 1-3/4 in.)

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Fig. 5.12 Stress, Thrust, Moment, and Deflection, Test B-1A (Z = O in.)
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Fig. 5.13 Stress, Thrust, Moment, and Deflection, Test B-1B (Z = O in.)







Fig. 5.15 Stress, Thrust, Moment, and Deflection, Test B-2 (Z = 7/8 in.)

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Fig. 5.16 Stress, Thrust, Moment, and Deflection, Test B-3 (Z = 1-3/4 in.)

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Fig. 5.18 Stress, Thrust, and Moment, Test B-6 (Z = O in.)

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Fig. 5.19 Stress, Thrust, and Moment, Test B-7 (Z = 7/16 in.)





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Fig. 5.24 Stress, Thrust, Noment, and Deflection, Test C-4 (Z = 3/16 in.)



Fig. 5.25 Stress, Thrust, Noment, and Deflection, Test C-5 (2 = 5/16 in.)



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Fig. 5.29 Stress, Thrust, and Moment, Test C-7 (2 = 3/16 ir.)







Fig. 5.31 Stress, Thrust, and Moment, Test C-9 (Z = 7/16 in.)

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Fig. 5.33 Thrust, Moment, and Deflection, Test E-3 (Z = 0 in.)

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Fig. 5.34 Thrust, Moment, and Deflection, Test E-2 (Z = 7/16 in.)

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Fig. 5.35 Thrust, Moment, and Deflection, Test E-1 (Z = 7/8 in.)



Fig. 5.36 Strain, Test E-5 (Z = 7/16 in.) and Test E-4 (Z = 7/8 in.); Surface Overpressure = 250 psi



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Fig. 5.37 Thrust and Moment, Tests E-6 (2 = 0 in.), E-5 (2 = 7/16 in.), and E-4 (2 = 7/8 in.); Surface Overpressure = 250 psi



Fig. 5.38 Stress, Thrust, Moment, and Deflection, Test D-1 (Z = O in.)



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Fig. 5.39 Stress, Thrust, Moment, and Deflection, Test D-2 (Z = 7/16 in.)

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Fig. 5.40 Stress, Thrust, Moment, and Deflection, Test D-3 (Z = 7/8 in.)

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Fig. 5.41 Stress, Thrust, Moment, and Deflection, Test D-4 (2 = 1-3/4 in.)

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Fig. 5.42 Stress, Thrust, Moment, and Deflection, Test D-5 (Z = 2-5/8 in.)





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Fig. 5.44 Cylinders of Groups A, B, and C after Tests



Fig. 5.45 Cylinders of Groups D and E after Tests



Fig. 5.46 Cylinders D-6 and D-10 after Test







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156

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Fig. 6.1 A Group: Average Spring Line Thrusts, Crown Moments, Vertical Diameter Changes, and q Values


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Fig. 6.2 B Group: Average Spring Line Thrusts, Crown Moments, Vertical Diameter Changes, and q Values









Fig. 6.4 Peak Diameter Changes and Deflection Stiffnesses

 
 STATIC TEST
 E

 C-1
 0

 C-3
 3/46

 C-2
 7/46

 C-3
 7/46

 D-1
 0

 D-2
 7/46

 D-3
 7/46

 D-5
 2-5/46

 E-3
 0

 E-2
 7/46

 E-1
 7/46
 STATIC TEST A-1 A-5 A-2 A-3A A-4 B-1A B-1A B-5 B-2 B-3 B-4 <u>Synelot</u> 0 ... ... ... ... ... LINE. 20060. 4 ... ... ... ... ... ... 0 3/16 7/16 7/8 1-3/4 0 7/16 7/8 1-3/4 2-5/8 AR = N<sub>y</sub> PloR 1.4 1.3 1.2 MASSIVE OR 1.1 1.0 ACTIVE ) OR } 0.9 D RATIO, AN 0.7 STATIC 0.0 0.5 c-0-Š 0.4 0.3 0.2 0.1 NCE OVER STATIC SING Fig. 6.5 Static Arching Ratio

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Fig. 6.8 Relation Between Nondimensional Pressure and Equation 3.16

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Fig. 6.9 Relation Between Nondimensional Pressure and Equation 3.8

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Fig. 6.10 Relation Between Nondimensional Pressure and Equation 3.17

167

#### APPENDIX A. PROPERTIES OF ALUMINUM TUBE MATERIAL

The cylindrical test specimens were cut from 12-ft lengths of Alcoa, drawn, aluminum tubing which was commercially available. Static, mechanical properties for the material are published in the manufacturer's literature, Aluminum Company of America (1960, p 59, and 1962, p 162). However, the values given are either minimum or average values, and hence do not adequately describe a given piece of tubing. Additionally, it was necessary to know the full stress-strain curve for the material up to the maximum strains experienced during the cylinder tests. In many cases, these strains far exceeded the indicated yield values.

Longitudinal tension test specimens were cut from each end and from the center section of the 12-ft lengths of tubing. The specimens averaged about 10 in. in length and were proportioned in accordance with ASTM Designation: E8-61T, <u>ASTM STANDARDS 1961</u>, Part 3 (pp 165-181).

The flat grips of the tension test machine proved unable to hold the slightly curved test specimen adequately once yielding commenced. Therefore, a special adapter was designed to accommodate the curvature of the specimen to the flat test grips.

Specimen from the tubes designated A, B, and C were all tested at the University of Illinois in a Tinius Olsen Testing Machine. It was used as a constant strain-rate device. An average crosshead speed of 0.05 in./min was used. It was first thought that the strain could be recorded adequately by monitoring with a manually operated Baldwin strain indicator. This proved satisfactory only for strains below first yield. The strain indicator operator was not able to keep a continuous balance

above yield due to the large strain changes. Hence, the system finally established utilized a Moseley X-Y plotter to record both load and strain simultaneously.

Specimens from the tubes designated D and E were tested at WES in a 30,000-lb, Riehle universal testing machine. An X-Y plotter was again employed.

Average stress-strain curves were developed for each 12-ft tube. They are plotted in Fig. A.1 and reduced to a finite number of digitized points in the tables shown on the figure. These points were used to describe the curve for the computer program.

The tension tests revealed no systematic variation in stressstrain characteristics along the length of the 12-ft tubes. The modulus of elasticity for the material,  $10 \times 10^6$  psi ( $\pm 5\%$ ), was verified by all of the tests. However, the inelastic stress-strain curves for the 6061-0 A, D, and E material varied from the average by  $\pm 10$  percent. The overall accuracy of the measurements, procedure, and reduction of data "or the 6061-T6 and 5052-0 material was within  $\pm 5$  percent of the average.

Although tubes A, D, and E were made of the same material, 6061-0, the inelastic stress-strain properties were sufficiently different from tube to tube that separate stress-strain curves were utilized in the data reduction.

Handbook yield values taken from Aluminum Company of America (1960, p 59) point up the fact that all the tubing does, in fact, exceed the manufacturer's indicated strengths. The values are indicated in Fig. A.1.

The stress-strain properties in Fig. A.1 were used in the

computation of thrust and moment under both static and dynamic loading. It was assumed that the static stress-strain relation would be a good approximation of the dynamic stress-strain relation. Aluminum is not, in general, strain-rate sensitive according to Steidel and Makerov (1960) and

Smith (1963).



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Fig. A.1 Aluminum Stress-Strain Properties

## APPENDIX B, PROPERTIES OF SANGAMON RIVER AND COOK'S BAYOU SANDS

### B.1 Placement Techniques

Since special effort was made to place and control the quality of the soil medium, it is probably denser and more uniform than that which could be obtained in a field installation.

#### B.1.1 Sangamon River Sand

This sand was stored in closed 55-gal drums near the testing device. A 2-gal water bucket was filled with sand, weighed on Toledo scales (0.1-1b graduations), carried to the test device and sprinkled into place. The sand was placed in 6-in. lifts. After 6 in. of sand had been placed, the lift was vibrated with a probe-type, concrete vibrator (Viber Co., Model II). The probe was vibrated completely through the 6-in. lift and was positioned on 2-in. centers in an ever-decreasing spiral around the center. This process was repeated until the test device was filled (four lifts) and screeded off.

A trench was then scooped out of the center of the sand and the cylinder placed at its intended depth and leveled. The sand was backfilled in the vicinity of the cylinder in 3/4-in. lifts by sprinkling the lift in and then rodding with a small ruler and tamping with a piece of wood.

The weight of sand displaced by the cylinder placement and subsequent backfilling was measured for each test. By assuming an effective volume of soil to be disturbed during placement, it was possible to calculate the average density of the sand in the immediate vicinity of the cylinders. The calculations indicated an average density of  $105.4 \pm 1.5$ pcf. The horisontal stiffness calculations in Section 6.2 also verify the

fact that the sand was very stiff. Penetration tests were not run because of the likelihood of disturbing the cylinder specimen. Additionally, recent research at WES\* on the use of penetration tests in dense sand has indicated that inherent scatter in hand-operated penetration test data within a layer is so great that variations in density on the order of 1 or 2 pcf are effectively masked.

The overall density was established by dividing the measured weight of the sand placed by the known volume of the test device (less the cylinder volume). The overall density was very reproducible and averaged 104.0 pcf with a minimum of 103.5 pcf and a maximum of 104.5 pcf.

The strain gages were continuously monitored during the placement. The A and B groups of cylinders were insensitive to the placement, but great care had to be exercised in backfilling around the very thin C group. In all cases the tendency was for elongation of the vertical axis. However, impressed strains were kept below 50 µin./in.

B.1.2 Cook's Bayou Sand

This sand was stored in piles on the floor and shoveled into the hopper of a sprinkling (also known as raining or showering) device. The gross weight was measured by an electric load cell. The sprinkling device was placed over the SELG base and maintained a known distance above the surface (24 in.); while the device was slowly turned at a constant rate, the sand dropped through the hoses, Fig. B.1. A density-height of fall study was made to determine an optimum turning rate and height of

Private communication with J. G. Jackson, Jr., Chief of the Impulse Load Section, Soils Division, U. S. Army Engineer Waterways Experiment Station, CE, Vicksburg, Miss., April 13, 1965.

fall.\* The full merits of the sprinkling technique are discussed by Whitman and others (1962, Appendix B).

The sand was placed up to the proposed level of the bottom of the cylinder. The cylinder was then positioned and sand was sprinkled in a manner intended to duplicate the free-field placement to bed the lower portion (90 degrees to 270 degrees). A piece of cardboard was used to deflect the sand beneath the spring line. The sprinkler was then repositionel and the remainder of the sand placed. The excess was screeded from the top to form a flat surface. A study\*\* to determine the effect of sprinkling sand around a small-scale buried structure has shown that the density in the vicinity of the structure can be about 2 pcf less than the average density in the free field.

The average density was  $106.6 \pm 1.0$  pcf. There was more scatter in average density with the sprinkling technique than with the vibration technique used for the Sangamon River sand.

B.2 Soil Strength and Deformation Characteristics

B.2.1 Sangamon River Sand

This sand was obtained from the Pontiac Stone Company, Mahomet, Illinois. It was wet and not of desired gradation when received. A system outlined by Prakash (1962, p 223) was used to obtain a uniform sand comparable to that tested by Hendron (1963). The sand was spread on the floor of

\* W. J. Turnbull, Chief, Soils Division, WES, "Soil Tests on Sprinkled Cook's Bayou No. 1 Sand Small Blast Load Generator Specimens," Memorandum for: Chief, Nuclear Weapons Effects Division, July 22, 1964.
\*\* R. W. Cunny, Chief. Soil Dynamics Branch, Soils Division, WES, "The Effect of an 8-In.-Dismeter Arch on the Density Produced by Showering Placement Method," Memorandum for: Chief, Muclear Weapons Effects Division, December 1, 1964.

the Illinois civil engineering test track and dried. Then, 8-lb batches were subjected to 5 min of sieving on a Gilson shaker (Model CL-262) that was fitted with a No. 40 and No. 60 sieve. The material retained on the No. 60 sieve was utilized for this investigation. The grain size distribution is shown in Fig. B.2.

The static, stress-deformation characteristics in triaxial and consolidometer tests are presented in Fig. B.3. These tests were run on sand having a density as close as possible to the overall average density used during the cylinder tests. The relative density,  $D_r$ , is also listed in Fig. B.3. Standard procedures were used.\* Moduli and shear strength data are presented in Fig. B.5.

B.2.2 Cook's Bayou Sand

This sand is commonly used in most of the WES blast load generator experiments, e.g. Tener (1964). It was procured locally and its characteristics were originally documented (then called Bayou Pierre Sand No. 1) in a WES Soils Division Memorandum for Record.\*\* However, recent laboratory tests performed for this investigation, Figs. B.4 and B.5, indicate that the one-dimensional stress-strain curve and angle of internal friction for a density of 106 pcf in the memorandum were in error.

It is evident that the two sands used have nearly identical laboratory properties at the densities employed because they were placed at equal relative densities. Also, the differences in the techniques used

\* Described in a laboratory manual prepared by the Waterways Experiment Station for the Office, Chief of Engineers, which will be issued as a Corps of Engineers Engineer Manual.

<sup>\*\*</sup> P. F. Hadala, Impulse Load Section, Soils Division, WES, "Soils Laboratory Tests on Bayou Pierre Sand No. 1," Nemorandum for Record, 1963.

to place the sand in the vicinity of the cylinder negate any refinements in explaining differences in laboratory soil properties. The sand around the cylinder in the Cook's Bayou sand tests may have been only of medium relative density.

## **B.3** Elastic Properties

Hendron (1963, p 84) concluded that the coefficient of earth pressure at rest,  $K_0$ , varies inversely with the angle of internal friction,  $\not o$ , as determined from drained triaxial tests.

$$l_{\mu} = 1 - \sin \beta \qquad B.1$$

For these sands,  $p = 36^\circ$ , Fig. B.5, and therefore

$$K_0 = 1 - \sin 38^\circ = 1 - 0.6 = 0.4$$
 B.2

If the soil were an elastic medium,

$$K_0 = \frac{v_8}{1 - v_8} \qquad B.3$$

and hence

$$v_{\rm g} = \frac{K_{\rm o}}{1+K_{\rm o}} = \frac{0.4}{1.4} = 0.19$$
 B.4

where v is Poisson's ratio for the soil.

Young's modulus of elasticity for the soil,  $\mathcal{B}_{s}$ , may be expressed in terms of the constrained modulus from t e consolidation test,  $M_{c}$ , as

$$E_{s} = \frac{(1 + v_{s})(1 - 2v_{s})}{(1 - v_{s})} M_{s}$$

$$= \frac{(1 + 0.29)(1 - 0.58)}{(1 - 0.29)} M_{s}$$

$$= \frac{(1.29)(0.42)}{0.71} M_{s} = 0.76 M_{s}$$
B.6

Variations of the constrained secant modulus,  $M_{CS}$ , with vertical pressure are plotted in Fig. B.5. Une-dimensional properties obtained at

175

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several relative densities for the Cook's Bayou No. 1 sand are reported by McNulty (1965).

Whitman and Healy (1962), discussing triaxial test results, and Davisson (1964), discussing one-dimensional test results, have indicated that essentially no dynamic strain-rate effects exist for dense, dry sands of the type used in this investigation.



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Fig. B.2 Gradation Curves for the Sands

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Fig. B.3 Stress-Strain Relations for Sangamon River Sand



Fig. B.4 Stress-Strain Relations for Cook's Bayou No. 1 Sand

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#### APPENDIX C. PROPERTIES OF BUCKSHOT CLAY

## C.1 Placement Techniques

The placement of small-scale test structures in a clay soil is a new endeavor. Luscher and Höeg (1964, pp 219-225) pointed out some of the difficulties in soil control. Inherent in the WES test setup are two additional difficulties: (a) the cylinder ends are closed before burial so that strutting the diameters is impracticable; (b) the cylinder cannot be positioned vertically for soil placement and then positioned horizontally for loading because the test chamber is a complete ring and cannot be sectioned.

It was therefore decided to use a technique already developed for footing tests, Carroll (1963) and Jackson and Hadala (1964), to place and compact the clay to the top of the 2-ft-deep test device. The cylinder would then be placed in the medium by cutting out a trench of appropriate dimensions in the center of the clay specimens and carefully backfilling around the cylinder. The technique for accomplishing the latter task was determined and patterned after procedures described in a feasibility study.\* Although adequate for the present investigation, the technique still has some drawbacks which will be discussed below.

The procedure followed in placing the 2-ft-thick clay specimen in the SBLG ring is shown in Fig. C.1. The clay was mixed in a pugmill and brought to the test area by truck, Fig. C.1a. When stored, the clay was kept continuously sealed in a polyethylene membrane (wrapper) except when

181

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<sup>\*</sup> R. W. Cunny, Chief, Soil Dynamics Branch, Soils Division, WES, "Tentative Placement Technique for Cylinders Buried in Clay Specimens," Memorandum for: Chief, Nuclear Weapons Effects Division, 1965 (in preparation).

soil was removed. The soil was processed on Fridays and allowed to cure over a weekend. The soil was weighed so that each loose lift, Fig. C.lb, would produce a 2-in. compacted lift. The loose soil was first hand-tamped, Fig. C.lc, and then compacted by three passes of a pneumatic tamper, Fig. C.ld. The soil surface was scarified, Fig. C.le, between lifts. The quality of the placement was controlled and chercked primarily through the use of density samples, Fig. C.lf. Vane-shear tests, Fig. C.lg, were made for certain specimens. Unconfined and confined compressive tests were performed on soil cubes that were taken from the top 8 in. of the specimen before and after each test. The pretest cube was taken at a distance of 1 ft from the cylinder and the hole was filled in a manner to duplicate the free-field placement. These results, as well as water content and density determinations, are given in Table C.l.

The cylinder was placed by cutting out an area in the center of the 2-ft-thick clay specimen, Fig. C.2a. The length and width of the cavity were the same for all tests and only the depth was varied. A template was used to size the excavation, Fig. C.2b, and it also served as a guide for the scooping operations, Fig. C.2c, which cut out a seat for the bottom half of the cylinder. The half-cylindrical cavity was formed exactly to the cylinder dimensions, and areas were carved out to accommodate the strain gages and end nuts, Fig. C.2d. The cylinder was then placed in the carved-out area, Fig. C.2e. The backfill was placed in loose, 3/4-in. lifts, Fig. C.2f, and compacted by three passes of a Harvard miniature compactor, Fig. C.2g. A lift is shown in place in Fig. C.2h. It is believed that very close contact was achieved between the clay and the cylinder.

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All 16 hoop strain gages were monitored during the placement operation. Some strain was impressed into the cylinder during each phase of the placement. Although several remedial methods (such as imposing a small vertical load on the cylinder through a saddle adapter) were tried to eliminate the strains, it was only possible to minimize them. About 40 percent of the total strain caused by placement occurred during the first seating phase, Fig. C.2e. Much of the remainder came during the first and second backfilling lifts; very little disturbance was noted in the cylinder due to lifts placed after the crown was covered.

The strains were primarily bending in nature and were most severe at the quarter points. The strains indicated that the cylinders assumed a slight vertical-elliptical shape. They probably did not significantly influence the failure pressure or mode of failure.

The average impressed strain resulting from the placement is shown in Fig. C.3. It is apparent that this placement technique must be improved before it can be applied to more flexible cylinders. Dorris and Albritton (1965) had very satisfactory results with this technique on a stiffer cylinder ( $EI/R^3 = 82$ ).

The placement technique was tedious and required a considerable amount of time. So much hand labor is involved that each of the ten tests required an average of one week in the testing device. Great care had to be taken to keep the clay sealed to avoid moisture loss. The water content determination from the cube tests indicates that this was successful (Table C.1).

The placement technique in the WES laboratory is probably better than that which could be achieved in a field installation.

# C.2 Soil Strength and Deformation Characteristics

The gradation curve and specific gravity,  $G_s$ , are shown in Fig. C.4. The clay is classified as a CH, and the results of several Atterberg limits tests are shown in Fig. C.5. The static, unconfined compressive strengths,  $q_u$ , were determined in the laboratory from samples taken from 8-in. cubes cut from the in-place clay specimens (the hole was refilled to the same density). The results are plotted in Fig. C.6. Average values are listed for each test in Table C.1.

In order to establish the quality of the backfill, specimens of clay were compacted in a mold in as nearly the same manner as the backfill was compacted. Unconfined specimens were cut from the mold and tested. The results are plotted in Fig. C.6. Those, coupled with the information from the vane-shear tests, indicate that the backfill was about 25 percent weaker than the compacted soil in the free field. Some of the weaker mold specimens were honeycombed (visual inspection) and this resulted in the lower values plotted in Fig. C.6 and the lower density values plotted in Fig. C.9. These molds were made during the early weeks of the investigation, and they may not have been truly representative of the compacted backfill.

Static triaxial (UU) test results are plotted in Fig. C.7. The degree of saturation,  $S_r$ , was about 90 percent and an apparent friction angle,  $\emptyset$ , equal to 1.7 degrees was deduced.

In order to establish an approximation to the one-dimensional stress-strain relation, three consolidation tests were run in which the vertical stress was applied and the deformation recorded as a function of time, Fig. C.7.

Moduli from the triaxial (UU) tests are plotted against confining pressure,  $\sigma_3$ , in Fig. C.8. These moduli exhibited a negligible increase with confining pressure and a line representing the average value is shown. Also in Fig. C.8, the secant modulus from the consolidation test at 6 sec elapsed time (after load application) is plotted with respect to vertical pressure.

The dry density,  $\gamma_{\rm d}$ , is shown in Fig. C.9 with respect to water content. It can be seen that the in-place soil is very similar to that used by Jackson and Hadala (1964).

Carroll (1963) conducted dynamic triaxial tests on buckshot clay (w = 27.1%) and indicated that the clay is strain-rate sensitive. However, the dynamic cylinder tests of this investigation either masked the effect or did not benefit from it. Kane and others (1964) discussed the behavior of clay under rapid and dynamic loading.

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Table	

Pretest Properties of Clay Specimens

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					-			·	Averag	e Dry t	Avera	ge UC				
			Avernce	Water	Content	i. V. Perc	ent	Cube Teste.	Weight	s, pcf	3	- 68 -	Av		Pressure	Average Posttest
	Ċ					From Top	Prom	Degree of	8 10-	Soll	Stre	ngth	Vane	strength	Applied	UC Com-
Test		rder of struction Date	Requested from Pur Mill	Pug Nilia	In Truck	0 in. Density Samples	Soil Cube Tests	satura- tion, S <sub>r</sub>	of Ring 7.4	Cube Tests Ya	Soll Soll Soll Soll Soll Soll Soll Soll	Nold	tons/ Soil Field	Backfill	During Cyl. Test Dei	pressive Strength tons/sq ft
<b>D-1</b>	-	11/16/64	26	25.2	24.9	24.9	25.2	89.9	96.3	96.0	2.90	1.65	3.97		8	3.8
<b>D-</b> 2	2	11/30/64	25	24.7	24.7	24.1	24.2	91.3	97.3	96.1	3.16	1	1	١	130	3.6
D-3	ŝ	<del>1</del> 9/1/21	25	24.7	24.7	23.8	23.9	87.2	97.8	6*96	2.83	ł	4 1	:	100	8°†
<b>D-4</b>	न	<b>₩9/₩I/ZI</b>	25	25.2	25.4	24.5	т° †г	88.9	96.5	96.7	2.72	2.22	ł	1 1	190	3.9
D-5	ŝ	<del>1</del> 9/12/21	25	25.2	25.4	24.3	24.3	90.2	9.96	91.5	3.38	1.89	1	;	180	ग∙ग
D-10	9	1/11/65	25	25.5	24°7	24.1	24.0	88.5	98.1	97.3	3.05	ł	1	8	97	3 <b>.</b> 4
6-0	7	1/18/65	25	25.5	24°1	23.7	23.7	89.1	∂ <b>.</b> 6	98.1	3•35	1.91	5.28	4.22	148	3.6
<b>D-8</b>	Ø	1/25/65	25	27.0	25.0	25.6	25.0	88.2	95.3	95°4	2.52	3.10	4.20	ł, 06	911	2.9
D-7	6	2/1/65	25	27.0	25.0	26.1	26.7	90.7	95.0	93.8	1.9	2.18	:	1	180	2.1
<b>D-6</b>	5	2/8/65	25	1	ł	2h.6	24.3	0*68	6•96	6•96	3•36	2.76	1	1	160	3.1
				× ×	verage		24.6	89.3	96.8	96.7	2.89	2.24	84.4	41.4		

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Fig. C.4 Gradation Curve for Buckshot Clay



Fig. C.6 Unconfined Compressive Strength-Water Content Relation





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Fig. C.9 Density-Moisture Content Relation for Buckshot Clay

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#### APPENDIX D. TRANSDUCERS

## D.1 Strain Gages

Gages 3/8 in. long were used because gages long enough to give reasonable average strains but short enough to eliminate the necessity of making curvature corrections were desired.

However, to initiate the investigation while the 3/8-in. gages were being procured, cylinders B-1 through B-5 were instrumented with 1/4-in.-long gages. They were Budd Metalfilm strain gages, Type C12 141B, 1/4 in. by 1/8 in. The remainder of the B group and all of the A and C groups were instrumented with Type C12 161. These gages were all temperature-compensated for aluminum. They are not classified as postyield strain gages but are capable of measuring strains accurately to 4-5 percent, according to the manufacturer. The gages functioned satisfactorily on tension test specimens (Appendix A) in that they measured strains accurately to values greater than 2 percent, and appeared to perform satisfactorily for the cylinder measurements.

Procurement complications prevented the acquisition of identical gages for cylinder groups D and E. Instead, Baldwin-Lima-Hamilton gages, Type FA-37-12-S13, were used. These are also 3/8-in. gages which are temperature-compensated for aluminum. The manufacturer indicates that these are accurate to 2 percent strain and they performed satisfactorily on tension test specimens strained beyond 1-5/10 percent.

Eastman 910 comment was tried as a gage adhesive on several tension test specimens, but was found to be unsatisfactory for strain levels beyond 0.5 percent. Armstrong adhesive C-2 was used to bond
all of the strain gages to the cylinders.

The inside gages were waterproofed by an application of Gagekote No. 1 (a solvent-thinned synthetic resin compound) while the outside gages were covered with Gagekote No. 5 (a two-compound, rubber-like epoxy resin) followed by Gagekote No. 2 (a solvent-thinned nitrile rubber) to isolate them further from the soil media.

A limited study was made to determine the potential influence of the soil pressure (acting as a normal force) on the outside strain gages. Four gages were mounted on a piece of 1/2-in.-thick aluminum plate and covered with various protective coatings, Fig. D.1. Gage 1 had a metal cover so that no soil pressure could reach the gage, and hence it served as a control on the response of the other three gages. All gages were waterproofed. Gage 2 was covered with a 0.015-in.-thick strip of fishpaper, gage 3 with Gagekote No. 5, and gage 4 with a piece of electrical, rubber tape. The plate was horizontally buried in sand and loaded statically to 300 psi. Negligible differences were noted in the response of the four gages, and the technique used for gage 3 was selected for its ease of use.

## D.2 Diameter Change Gages

A diameter change gage was required which would be expendable since the cylinder collapse would destroy anything inside. The transducer used was recommended by Professor V. J. McDonald of the University of Illinois. It consisted of a curved strip of 0.01-in.-thick brass shim stock 1/4 in. by 6 in., Fig. D.2. Budd Netalfilm, Type Cl2 141B, strain gages were mounted on each side of the strip's center with Eastman 910 cement. The gages were joined electrically to indicate only the bending

194

strains. Two 1/32-in.-diameter holes were drilled in each end of the strip and in the cylinder. The same nut and bolt arrangement was used for mounting the strip in the cylinder, Fig. 4.1a, as was used in calibration.

Each diameter change gage was calibrated in extension and compression in a Pratt and Whitney Super Micrometer. The apparent strain gage output was a linear function of displacement, and amounted to  $5 \,\mu in./in.$ per 0.001 in. of diameter change.

The gage could not be used for rapid or dynamic testing because it experienced excessive ringing under these loadings. Gages were coated with petroelastic to dampen the spurious vibrations but no improvement resulted.

D.3 Overpressure Gages

For the tests conducted at the University of Illinois, Bourdon gages were used to measure the static overpressure. Their accuracy was verified relative to other available gages.

The rapid pressure tests were monitored by a Kistler piezoelectric pressure transducer Model 601. The transducer was calibrated prior to testing and its output was a linear function of overpressure, 0.41 picocoulombs per psi or about 125 psi per inch of paper deflection. The gage was checked after each test, and no calibration changes were required.

Both the static and dynamic tests at WES were monitored by Norwood pressure transducers Model 211. These were statically calibrated prior to each test, and exhibited a generally linear response. They were ranged for about 250 psi/in. of paper deflection statically, and 125 psi/in. of paper deflection dynamically.

195

At least two gages were used in each test and the measured pressure for the gages was within ±5 percent of the sverage. A Bourdon gage was used to verify the peak static pressure and thereby made the static values more reliable; but the dynamic results probably varied either because of the use of a static calibration or because of the motions of the gage mounts. The gages were located between the firing tubes in the dynamic bonnet. A study by Kennedy and Sadler (1965) has shown that the surface pressure distribution is uniform within ±10 percent.

196



Fig. D.1 Strain Gage Test





Albert Francis Dorris was born in Utica, New York, on October 25, 1936. Following graduation from Utica Free Academy in 1955, he entered the United States Military Academy. He was named to the 1957 All-American Trok and Field Team. In his senior year, he was appointed a Cadet Captain. He received the Bachelor of Science Degree in June 1959 and stood 6th in graduation order of merit cut of a class of 499. Upon graduation he was commissioned as a regular officer in the United States Army, Corps of Engineers. He attended the Engineer Officers' Basic Course at Fort Belvoir, Virginia, and the U.S. Army's Airborne and Ranger courses at Fort Benning, Georgia, enroute to his initial assignment in Korea. From June 1960 to June 1961 he was with the 547th Engineer Company (Float Bridge) in Korea as a Platoon Leader and Company Commander. In August 1961 he entered the Engineer Officers' Advanced Course at Fort Belvoir and subsequently worked for a brief period in the Office of the Chief of Engineers before entering graduate training in Civil Engineering at the University of Illinois in June 1962. He received the Master of Science Degree from Illinois in June 1963 and from June 1964 to June 1965 was assigned to the U.S. Army Engineer Waterways Experiment Station, Vicksburg, Mississippi, as a Projec+ Engineer. He currently holds the rank of Captain in the Corps of Engineers and is a member of the American Society of Civil Engineers, the Society of American Military Engineers, and the Association of the United States Army. He is an Engineer-in-Training in the State of Illinois.

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