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TECHNICAL REPORT NO. 1-682

# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

by

Albert F. Dorris



Sponsored by

Defense Atomic Support Agency

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Conducted by

U. S. Army Engineer Waterways Experiment Station CORPS OF ENGINEERS

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July 1965

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Vicksburg, Mississippi

ARMY-MRC VICKSBURG, MISS

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#### FOREWORD

This report was prepared in the Nuclear Weapons Effects Division, U. S. Army Engineer Waterways Experiment Station, under the sponsorship of the Defense Atomic Support Agency (DASA) as part of NWER Subtask 13.010, Response of Buried Structures to Ground Shock. The work was accomplished during the period February 1964 through May 1965. During this time, Mr. G. L. Arbuthnot, Jr., was Acting Chief of the Nuclear Weapons Effects Division, and Mr. W. J. Flathau was Acting Chief of the Protective Structures Branch.

This report was prepared by Captain Albert F. Dorris, CE, and is essentially a thesis submitted in partial fulfillment of the requirements for the degree of Doctor of Philosophy in Civil Engineering to the University of Illinois, Urbana, Illinois.

Directors of the Waterways Experiment Station during the period of this study were Colonel Alex G. Sutton, Jr., CE, and Colonel John R. Oswalt, Jr., CE. Mr. J. B. Tiffany was Technical Director.

#### SUMMARY

This was an experimental investigation into the response of small, shallow-buried (in dense, dry sand and stiff clay), aluminum cylinders to static (15-min rise time), rapid (13 msec), and dynamic (0.3 msec) plane-wave loading up to 500 psi. Th. cylinders had identical outside diameters of 3.5 in. and two thicknesses, 0.022 and 0.065 in. Hence, the cylinder stiffnesses,  $EI/R^3$ , were 1.7 and 45 (d/t = 159 and 54), respectively.

In stiff clay, the overpressure required to cause collapse increased very slowly with increasing depth of burial from zero to the deepest burial, three-quarters of the diameter. The hydrostatic buckling equation,  $P_{cr} = 3 EI/R^3$ , was applicable for the cylinders tested.

In the dense id, the overpressure required to cause collapse increased greatly with increasing depth of burial from zero to one-eighth of the diameter. Below this depth it was not possible to collapse even the most flexible cylinders under the available 500-psi pressure. The hoop compression theory was verified. A ductility factor of about 7 was found to be conservative for cylinders buried at depths greater than oneeighth their diameter in the dense sand.

The recorded strains were nonelastic in many cases and it was shown that large yielding does not necessarily define collapse. Stress and moment were found to be nonlinear functions of overpressure, whereas thrust was generally found to be a linear function of overpressure. The differences between static and rapid loading in the elastic response of the cylinder were found to be small. Diameter changes recorded prior to collapse for the static tests were small, less than 5 percent of the diameter.

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THE GRADUATE COLLEGE

May 11, 1965

I HEREBY RECOMMEND THAT THE THESIS PREPARED UNDER MY

SUPERVISION BY\_\_\_\_ALBERT FRANCIS DORRIS

ENTITLED\_\_\_\_\_RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS

TO STATIC AND DYNAMIC LOADING

BE ACCEPTED IN PARTIAL FULFILLMENT OF THE REQUIREMENTS FOR

THE DEGREE OF \_\_\_\_\_ DOCTOR OF PHILOSOPHY IN CIVIL ENGINEERING

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s/ N. M. Newmark

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† Required for doctor's degree but not for master's.

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# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

BY

ALBERT FRANCIS DORRIS B.S., United States Military Academy, 1959 M.S., University of Illinois, 1963

# THESIS

Submitted in partial fulfillment of the requirements for the degree of Doctor of Philosophy in Civil Engineering in the Graduate College of the University of Illinois, 1965

Urbana, Illinois

#### ACKNOWLEDGMENT

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This thesis is based upon experimental studies conducted at the University of Illinois and the U.S. Army Engineer Waterways Experiment Station (WES). The tests conducted at Illinois were sponsored by the Department of Civil Engineering.

These studies were conducted under the general direction of Dr. N. M. Newmark, Professor and Head of the Department of Civil Engineering, and under the direct supervision of Professor G. K. Sinnamon of the Department of Civil Engineering.

Acknowledgment is made to 1st Lt. A. J. Hendron, Jr., and Mr. W. J. Flathau for their comments and encouragement, and to Mr. W. H. Sadler, Jr., who assisted in all phases of the study at WES.

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### NOTATION

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a Radius of the intrados of the cylinder A Area of the cross section perpendicular to the ring center line AR Arching ratio b Radius of the extrados of the cylinder  $C_u$  Uniformity coefficient,  $D_{60}/D_{10}$ d Outside diameter of cylinder D<sub>r</sub> Relative density,  $\frac{e_{max} - e}{e_{max} - e_{min}}$ <sup>D</sup>10 Soil grain diameter of which 10 percent of the soil weight is finer Soil grain diameter of which 60 percent of the soil weight D60 is finer e Void ratio,  $\frac{V_v}{V_v}$ Maximum void ratio e<sub>max</sub>  $\mathbf{e}_{\min}$ Minimum void ratio Initial void ratio e o Modulus of elasticity of the cylinder, Young's modulus Е E' Modulus of soil reaction, equal to k R, psi E<sub>s</sub> Modulus of elasticity of the soil Acceleration of gravity g  $G_{g}$  Specific gravity of the solids Thickness of the cylinder wall h I Moment of inertia of the cross section of the cylinder wall per unit length, in.  $\frac{1}{y}$ 

k i	Spring	constant,	load	divided	by	deflection
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- ${\bf k}_{\,\rm a}$  Coefficient of elastic soil reaction, psi per strain
- k\_ Coefficient of soil reaction ("subgrade modulus")
- k Modulus of passive resistance of the enveloping earth, psi per inch of deflection, lb/in.3
- k, Radial elastic support
- K Coefficient of earth pressure at rest
  - & Cylinder length
- M Bending moment at the cylinder crown, constrained soil modulus
- M<sub>cs</sub> Constrained secant modulus of soil
- M\_ Bending moment, M
  - n Buckling mode number or order; number of half-waves
- N. Thrust or normal force in the cylinder, lb/in.
- p Pressure, psi
- p Vertical pressure on a horizontal plane through the cylinder crown

Pm,Pt

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- p Critical buckling pressure in lowest mode for a ring subjected to hydrostatic pressure.
  - P Vertical force, 1b
- P Overpressure on surface of soil, psi
- P Overpressure on surface of soil when cylinder collapsed
  - q Ratio of average horizontal force (or pressure) to average vertical force (or pressure) applied to the cylinder
  - q. Unconfined compressive strength
    - Q Vertical shear force in soil between surface and cylinder crown

- Q' Vertical shear force in soil between cylinder crown and spring line
- Q" Oblique shear force in soil between cylinder crown and spring line
  - r Radius of a cylinder element
  - R Radius of the cylinder middle surface
- S\_ Degree of saturation
- S,S, Relative stiffness
  - t Time
  - $\mathbf{T}_{\mathbf{x}}$  Period of vibration in the compressive mode
  - T, Period of vibration in the first flexural mode
  - TD Typical descriptor of relative stiffness
  - V Total volume of soil sample
  - V Initial volume
  - V Volume of soil solids
  - V Volume of voids
  - w Radial displacement of the cylinder; water content

x,y,z Cylinder coordinates, spatial cordinates

- Z Vertical distance from soil surface to cylinder crown
- 7 Unit weight of soil, specific weight
- $7_d$  Dry unit weight
- A Horizontal deflection (increase in diameter)
- A. Vertical deflection (decrease in diameter)
- **∆V** Volume change
- € Unit strain
- Strain on extrados of the cylinder
- e, Strain on intrados of the cylinder

- $\theta$  Circular angle
- v Poisson's ratio of the cylinder
- $v_s$  Poisson's ratio of the soil
- σ Stress
- $\sigma_{\mathbf{y}}$  Stress in the y or tangential direction
- $\sigma_{yl}$  Lower or first yield stress
- $\sigma_{y2}$  Upper yield stress (result in 0.2 percent permanent strain)
- $\sigma_1$  Vertical stress
- $\sigma_3$  All-around confining stress
- Ø Angle of internal friction

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# RESPONSE OF HORIZONTALLY ORIENTED BURIED CYLINDERS TO STATIC AND DYNAMIC LOADING

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CHAPTER 1. INTRODUCTION

### 1.1 Background

The art of designing buried structures to resist nuclear blast loading is still (1965) in its infancy. A desirable way of augmenting the development and evaluation of particular protective structures designs is to conduct full-scale tests; however, the moratorium on full-scale surface tests in effect since 1 November 1958 eliminates this approach in studying the response of shallow-buried structures to overpressure-induced disturbances. Unfortunately, even if full-scale tests had been permitted since 1958, it is doubtful that sufficient data would be available from such tests alone to formulate economical and practical designs for most design situations. Laboratory and analytical studies still would have been needed to supplement such programs. Because of the limitations imposed by the muratorium, special emphasis has by necessity been placed on analytical studies and laboratory tests of small-scale structures for the purpose of developing usable design methods.

At the moment there is a lack of well-documented experimental data and field experience with which to compare current thought and analytical theory. The most advanced design manual, <u>Principles and Prac-</u> <u>tices for Design of Hardened Structures</u> by Newmark and Haltiwanger (1962, under revision),\* and the current source book of underground phenomena and effects of nuclear weapons, <u>Nuclear Geoplosics</u> by Stanford Research

\* Authors and dates refer to list of references at end of text.

Institute (1964), point out a multitude of unknowns in the state of the art. 1.2 Problem Under Study

Buried cylindrical or ring configurations are ideal geometries to resist external loads effectively and are thus well suited to protect personnel and appurtenances for various facilities such as NIKE and ICEM hardened sites. They are also favored as entrances and escape routes for protective structures buried deep in rock. Additionally, almost all communication and utility conduits, existing and planned, are cylindrical in shape. Currently, these structures are being designed largely on the basis of engineering hypotheses supplemented by the field experience gained with buried conduits and tunnel liners subjected to static loading. There is virtually no experimental validation of the current dynamic design criteria. Because of the uncertainties, the current design procedures are only stopgap measures which await the results of controlled experimental investigations for confirmation or refutation.

The problems of designing shallow-buried protective s'ructures for overpressure-induced loading from large-yield weapons differ from those associated with other underground cylindrical structures in at least two major ways: (1) The live load is large compared with the dead load, and the structure must be designed primarily for the live load; (2) the criteria for failure, together with the factor of safety, must lead to the least expensive structure which couples cost and use to fulfill requirements. A factor of safety of 4 is common in culvert design as indicated by Armco Drainage and Metal Products, Inc. (1958, p 70). This factor is sufficiently large to take care of many unknowns. However, a factor this large is economically infeasible for the design of most protective structures.

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### 1.3 Objective of the Study

The objective of this investigation was to study experimentally the phenomena associated with the failure of horizontally oriented, circular cylinders buried at various shallow depths in several soil media and subjected to either static or dynamic overpressures.

# 1.4 Scope of the Investigation

It would be desirable to study a wide range of cylinder types by varying such parameters as material properties of the cylinder, cylinder dimensions, soil media, depths of burial, overpressure characteristics, and combinations of instrumentation transducers. Experimentally, very little ultimate strength work has been done to study buried cylindrical structures in the collapse range.

An evaluation of all the parameters and combinations in detail would be far beyond the scope of any single investigation. The parameters selected for study are outlined below:

- In order to examine the extreme range in soil media, two soils were selected: a dense, dry sand and a highly plastic clay placed at such a water content that the consistency ranged from stiff to very stiff as defined by Terzaghi and Peck (1948, p 31).
- In order to examine the effect of depth of burial,
   five shallow depths, ranging from zero to 2-5/8 in. or
   3/4 diameter (d), were investigated.
- 3. In order to examine overpressure effects, three pressure-time signatures were used, ranging from a quasi-static rise time of 10 to 15 min, to a rapid

rise time of 13 msec, and up to a dynamic rise time of 0.3 msec.

4. In order to examine a range in structural stiffness, two circular cylinder geometries (two wall thicknesses and three nominal yield strengths) were employed. The outside diameter, length, and end conditions were kept constant.

Since underground cy indrical structures have long been used as tunnels, culverts, sewers, and pipes, a great deal of qualitative knowledge is available covering all aspects of the soil-structure system, e.g. arching, longitudinal beam action, live load distribution, ring loading, and ring response. Fig. 1.1 illustrates some of the concepts of load transfer from the soil surface to the underground structure.

This test program was planned to investigate ring response, and the emphasis was not on the associated phenomena such as arching. These will be discussed only as they contribute to an understanding of the ring response.

Forty-six cylinders were tested during the investigation. For each rapid or dynamic test (plane wave loading), a corresponding static test was performed for comparison. The entire program is summarized in Table 5.1.

The 30 cylinders designated as groups A, B, and C were tested under static and rapid loading in the blast-loading facility at the University of Illinois. The 16 cylinders in groups D and E were tested under static and dynamic loading at the U. S. Army Waterways Experiment Station (WES).

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### CHAPTER 2. DEVELOPMENT OF PRESENT STATE OF KNOWLEDGE

### 2.1 Culvert, Pipe, and Tunnel Contribution

It is not the writer's purpose to cite all of the potentially applicable work, but rather to categorize the development of current schools of thought and to summarize the more pertinent references describing the development of design and analysis procedures for buried cylinders.

2.1.1 Talbot, Cain, Marston

Talbot (1908) tested cast-iron, plain concrete, and reinforced concrete pipes to failure. He recognized both the beneficial effect of lateral confinement (p 22) and the ability of the concrete rings to retain their circular configuration until final failure occurred when the concrete crushed (p 65). The idealized load distribution which he considered is shown in Fig. 2.1a. In view of the fact that the load distribution was not uniform, that the actual value of q (the average horizontal pressure divided by the average vertical pressure) was not determinable, and that cracking would not be acceptable for permanent installations, Talbot recommended the use of the formula  $M_c = 0.25p_aR^2$  for design, i.e. the maximum bending moment (at the crown),  $M_c$ , with q = 0 where  $p_a$  is the average pressure on a horizontal plane through the crown, and R the mean radius of the pipe. Any surplus strength offered by the side restraint would be "considered merely an additional margin of safety" (Talbot (1908)).

Braune, Cain, and Janda (1929) explored the possibility that the horizontal pressure was not distributed all the way to the top of the ring (Fig. 2.1b). Using the results of pressure cell measurements on the surface of relatively flexible rings, they (in Appendix II written by Cain)

tried to arrive at applicable values of  $\theta$ , the circular angle, and q. Cain also discussed (p 173) the reasons why deflections determined by a uniform radial load theory would never agree with measured values. This theory treats the horizontal passive soil resistances as if they were active soil forces.

Marston (1930) summarized his own work on arching and gave some guidelines to define the differences between flexible and rigid conduits. He considered flexible conduits as having cross-sectional shapes that can be distorted sufficiently to change their vertical or horizontal dimensions more than 3 percent before causing materially injurious cracks; rigid conduits cannot sustain such distortions.

2.1.2 Spangler

Spangler (1938) used a friction tape technique to measure the pressure distribution on the outside of flexible metal pipes. He developed a hypothetical distribution of pressure, Fig. 2.1c, based on the maximum unit horizontal pressure being equal to the modulus of passive resistance,  $k_g$ , of the fill material multiplied by one-half the horizontal diameter change. Spangler used e for this, but for distinction within this report the term  $k_g$  shall be used. He stated that deflection of a flexible culvert is the phenomenon of primary interest "because failure of flexible pipes occurs by excessive deflection rather than excessive stress." Spangler's design formula (Iowa Formula) for good bedding, Fig. 2.1c, also shows the relative influence of the pipe parameter, EI, and the influence of the passive soil resistance parameter, 0.061  $k_g R^h$ , where E is the modulus of elasticity of the pipe, I is the moment of imertia of the pipe wall, and R is the mean radius of the cylinder.

Spangler (1948) reviewed the state of knowledge of underground conduits and pointed out the lack of knowledge concerning the modulus of passive resistance,  $k_g$ . He also indicated that the load distribution on a horizontal plane at the level of the cylinder crown,  $p_a$ , is approximately uniform over the breadth of the pipe. Spangler (1956, pp 1054-9) discussed the validity of assuming a condition of plane strain or plane stress for pipeline problems. He concluded that it is not possible to determine which most nearly applies, and used the somewhat simpler plane stress assumption which is not dependent upon Poisson's ratio,  $\nu$ , of the cylinder. Spangler (1960, Chapter 25) further discussed the Iowa Formula and tentatively recommended that for flexible culverts the deflection should not exceed 5 percent of the diameter. Typical values for the modulus of passive resistance were mentioned. Spangler indicated that the modulus of passive resistance is strongly influenced by the size of the pipe and gave recommended values for design.

# 2.1.3 Watkins

Watkins and Spangler (1958) examined the Iowa Formula from a dimensional analysis or similitude point of view. It was concluded that the modulus of passive resistance is not a property of the soil alone; and, further, that the product of the modulus of passive resistance,  $k_g$ , times the pipe radius is a constant for a given soil. This quantity,  $k_g R$ , was termed the modulus of soil reaction, E'.

Watkins (1959) attempted to correlate the modulus of soil reaction to properties that are easily measured. His work indicated that the modulus was related to the compression index for a given soil. Watkins (1960) pointed to buckling of the pipe wall, before an excessive diameter

change has occurred, as a potential failure mechanism for buried conduit systems. Watkins (1963) suggested that the hydrostatic buckling equation,  $p_o = \frac{3EI}{R^3}$  (where  $p_o$  is the critical buckling pressure in psi), be applied as a conservative estimate of the buckling failure phenomenon. This and the work of Brockenburgh (1963) influenced the U. S. Steel Corporation to produce a new corrugation profile for their flexible culverts. Watkins and Nielson (1964) developed a test apparatus, modpares device, to measure the modulus of soil reaction. It was found that this quantity is not a constant, but rather decreases with increasing conduit deflection.

Watkins (1964) again pointed out the importance of the soil in influencing structural response, and illustrated the possibility of buckling for a very flexible ring carefully embedded in a well-compacted, granular fill.

2.1.4 Schafer, Barnard, White

Schafer (1948) stated that an average safe maximum deflection for conduits is 20 percent of the vertical diameter. Application of a factor of safety of 4 to the deflection criterion leads to a design deflection of 5 percent. He developed an empirical deflection equation, examined the lowa Formula, and concluded that it gave undue value to the side-support factor,  $k_g$ , for large-diameter structures.

Barnard (1957) pointed out that apparent bending stresses in steel pipe based on elastic theory are not of importance in themselves when the ductility of the material in the shell permits deformation without failure. Localized bending stresses which appear to pass the yield point of the material are not proper criteria for failure.

White and Layer (1960) proposed the ring compression theory,

Fig. 2.1d, as a rational design tool. They argued that the ring bending stiffness need only be sufficient (1) to prevent buckling, (2) to resist the uneven loads in minimum cover installations, and (3) to permit easy handling and erection. White (1961) described a 21-ft-diameter corrugated culvert designed by using the ring compression theory, and indicated that the primary factor for average corrugated metal conduits is compressive strength.

# 2.1.5 Meyerhof

Meyerhof and Baikie (1963) performed tests to failure on quarter sections of curved steel sheets bearing against dense sand backfill. They showed that for small values of the subgrade modulus and the flexural rigidity of the plates, the sheets would fail by buckling; but, for larger values of these parameters the sheets would fail by yielding of the section. The ring compression theory was supported. Their buckling theory, discussed in Chapter 3, indicates that the hydrostatic theory is overly conservative. Meyerhof and  $\Gamma$ 'sher (1963) discussed several field experiences and concluded that failures due to excess deflection were a consequence of unsuitable backfill material or poorly compacted soil. They urged the use of competent backfill so that the ring compression theory could be applied.

# 2.1.6 Large Field Structures

Terzaghi (1943) observed experimental sections of the Chicago subway tunnels in clay, and concluded that a nearly uniform distribution of pressure should be assumed. Terzaghi (1942, p 207) further suggested that the bending moments would be insignificant even in a fairly thick shell because the deformation of the tube automatically reduces the moments.

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Peck and Peck (1949) discussed observations made on largediameter, flexible steel culverts. They concluded that if the soil is adequately compacted, a moderate deformation will establish a state of nearly uniform all-arcund pressure.

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Lane (1960) described the observation made of tunnel test sections at Garrison Dam. In the flexible sections, the ratio of the horizontal to vertical load ranged from 0.8 to 1.1. However, higher bending moments were observed in the flexible ribs than could be explained by the small differences between the measured horizontal and vertical thrusts. Thus, the moment was apparently dependent on things other than the overall loading, such as the construction procedures.

2.2 Prote tive Structures Research

2.2.1 Dynamic Theory

A number of complex solutions have been generated for mathematical continuum models which are tractable within the classical theory of elasticity. Palmer and Lankford (1963) compared several solutions and recommended the approach taken by Yoshihara and others (1963) as being very promising. Albritton and others (1965) reported the results of an experimental pilot study of a stiff, buried cylinder and an extensive analysis of the mathematical and physical limitations of the currently available continuum theories.

Now (1964) reviewed various dynamic analyses and concluded that "under the assumption of earth media being elastic, homogeneous and isotropic, the dynamic-stress concentration factors for all cylindrical-cavity cases, whether elastically lined or unlined, are all about 10 to 20 percent higher than those for their corresponding static cases." The verification of this analytical prediction could reduce the problem (when a step pulse or instantaneously applied input assumption is applicable) to the simpler static case with an arbitrary 20 percent increase in design equations.

As a consequence of the work of Merritt and Newmark (1962) and Melin and Sutcliffe (1959), Newmark and Haltiwanger (1962) outlined the only theory known to the writer which takes into account the nonelastic behavior of the cylinder.

No directly applicable theory of dynamic buckling is known.

2.2.2 Static Theory

In addition to the mechanics' theories already mentioned in connection with culverts, Section 2.1, several possible elastic continuum theories exist. Palmer and others (1963) compared a number of these and suggested using the solution of Savin (1961) for a lined hole in an infinite plate. Other similar solutions can be found for the static case which evolve as limiting portions (longtime or steady state) of the dynamic analyses where they approach the static case.

2.2.3 Ultimate Strength Laboratory Tests

Bulson (1962) tested 56 thin tubes to failure under static loading up to 100 psi. Overpressure and dial deflections were the only measurements made, but these were sufficient to describe the failure mode as buckling. The failures at the deepest burial, 3/4d, in the dense sand point to a failure mode heretofore unrecognized for fully buried cylinders. Bulson (1963, a and b, and 1965) extended the work to square cylinders and (1964) summarized all of his previous tests.

Donnellan (1964) conducted nondestructive tests on instrumented

cylinders and destructive tests on noninstrumented cylinders buried in dense, dry, 20-30 Ottawa sand. The loading was quasi-static up to a maximum of 160 psi. Only the overpressure was monitored during the ultimate strength tests.

Whitman and Luscher (1962) and Luscher (1963) statically tested small aluminum tubes surrounded by dense sand and symmetrically loaded in a triaxial type device. As a result, Luscher and Höeg (1964) concluded that the major contribution of the sand to the system was to force the cylinder to respond in higher buckling modes. Luscher and Höeg (1964) also conducted buried tube tests which yielded failure conditions similar to those of the fully symmetric situation.

2.2.4 Nondestructive Laboratory Tests

A number of tests have been conducted to verify elastic theories and to form a basis for predicting the ultimate strength of a cylinder.

Allgood and Gill (1964) made a series of static and dynamic tests up to a maximum of 25-psi overpressure on a 24-in.-diameter steel cylinder buried in dense sand. All response was in the elastic range of the cylinder material. They found that the form of the deflection, thrust, and moment distribution was much the same under both types of loading. Some differences were noted: The maximum thrust under dynamic loading was about 14 percent higher than for static loading; the crown deflection under dynamic loading was about twice that under static loading. Allgood (1965), in attempting to summarize the case of a thin metal cylinder buried at shallow depths in a uniform, noncohesive soil, concluded that the net arching (reduction in vertical load below that at the surface) across a thin metal cylinder is negligible.

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Robinson (1962) presented the results of a series of static tests up to a maximum of 100-psi overpressure on 6-in.-diameter tubes buried in dense, dry Ottawa sand. Robinson (1964) extended the earlier tests by including more strain gages. Four test sections were used at a depth of burial of 15 in., 2-1/2d. The results were nonsymmetric in response and showed a great amount of scatter in the moments.

# 2.2.5 Full Scale Tests

Albright and others (1960) described the response of largediameter, buried conduit sections located at the 100-psi pressure range of Shot Priscilla (1957) in Operation PLUMBBOB, a full-scale field test. The sections were selected by means of modified static design procedures, and all survived the blast loading.

Williamson and Huff (1961) described the response of 20-ft long, 7-ft diameter, 10-gauge structural-plate pipes buried at a 10-ft depth of cover and subjected to a pressure of 250 psi from Shot Smoky of Operation PLUMBBOB. Again the structures survived with very small deformations and virtually no damage.

McDonough (1959) described tests on drum-shaped structural models buried at depths of from 0 to 20 ft and subjected to the effects of airinduced pressures resulting from large detonations. The compressibility of the structure relative to the surrounding soil appeared to govern the amount of load that was transmitted to the structure.

### 2.3 Similitude Studies

The American Machine and Foundry Company (1962) and Murphy and Young (1962) examined the feasibility of modeling the soil-structure interaction problem, and developed similitude relations. THE REAL PROPERTY AND A DESCRIPTION OF THE PROPERTY AND A DESCRIPTION OF THE

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Murphy and others (1963) demonstrated the feasibility of using small-scale modeling for qualitative results. Young and Murphy (1964) tested their similitude requirements on stiff aluminum cylinders buried in dry Ottawa sand, and concluded that the requirements were satisfied within the range of parameters investigated.

Dowell (1964) continued the work with stiff cylinders, but experienced difficulty as a result of sidewall friction in the testing device.

## 2.4 Bibliographies and Design Manuals

Van Horn and Tener (1963) and Merkle (1963) prepared annotated bibliographies on the subject of soil-structure interaction. Each chapter of the five volume set of <u>Nuclear Geoplosics</u> by Stanford Research Institute (1964) contains an excellent bibliography. <u>The Effects of Nuclear Weapons</u> by U. S. Atomic Energy Commission (1964) covers the general field of nuclear weapons, and the <u>Proceedings of the Symposium on Soil-Structure</u> <u>Interaction</u> by University of Arizona (1964) presents the most up-to-date research.

Design manuals appeared in 1957 with the U.S. Army Corps of Engineers series EM 1110-345-413 to -421. American Society of Civil Engineers (1961) and Newmark and others (1961) developed design recommendations. Newmark and Haltiwanger (1962, under revision) outlined design procedures for hardened sites.

#### CHAPTER 3. THEORETICAL CONSIDERATIONS

Various theoretical solutions and concepts are presented in this chapter and are compared with the test results in Chapter 6. The nonavailability of a dynamic buckling theory together with the theoretical indication that the dynamic response for a step pulse is only 10 to 20 percent greater than the static response suggests that static theory may be applicable for the elastic case.

### 3.1 Definition of Failure

A protective structure fails when it can no longer perform the function for which it was designed. For the shell under consideration, Fig. 3.1, failure is an inability to keep the ring from collapsing. This could come about by (1) the vertical diameter decreasing to such an extent, say 20 percent, that the crown would reverse curvature and plunge to the invert, Fig. 3.2a; (2) a section of the wall becoming unstable before a large-diameter change has occurred (and buckling inward into the cavity with a large amplitude) as a consequence of the interaction between thrust and moment (a) before any fiber in the cross section has yielded, (b) after some fibers have yielded in bending but before the whole cross section has yielded in thrust, (c) at some time after the whole cross section has yielded in thrust (hoop compression). Fig. 3.2b, c, and d show some observed modes of failure.

Large, i.e. greater than 5 percent, changes in diameter will not occur (if the cylinder is emplaced in a competent backfill) before one of the failure mechanisms in (2) above has triggered the structural collapse. The backfilling around protective structures should be carefully

controlled; therefore, the tests of the present investigation were conducted in well-compacted and -controlled sand and clay specimens.

Because the cylinder tends to readjust itself under load, it may be assumed that the bending moments are negligible in the development of a buckling criterion. Hence, failures (2)(a) and (2)(b) mentioned previously can be considered one condition describing the elastic membrane response of the cross section.

As long as the wall acts as a ductile member, yielding will not constitute failure other than as it precipitates inelastic buckling.

## 3.2 Elastic Buckling

### 3.2.1 Soil Medium Approximated by Water

A first approximation to the problem of a uniform soil-surrounded cylinder can be made by the use of the equation for hydrostatic buckling of a ring, Fig. 3.3. Since this mathematical model assume that the medium possesses no shear strength, it should serve as a lower bound for the buckling value for uniform radial loading. Seely and Smith (1952, p 612) arrived at the classical relation

$$p_{h} = (n^{2} - 1) \frac{EI}{R^{3}}$$
  $n \ge 2$  3.1

where  $p_h = uniform$  collapsing (critical) pressure (force per unit area)

# for the ring section

- n = buckling mode number, an integer
- E = modulus of elasticity of the cylinder material
- I = moment of inertia (per unit length) of the ring cross section
- R = mean radius of the ring

The minimum value for  $p_h$  , other than zero, is

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$$p_0 = 3 \frac{EI}{R^3}$$

Timoshenko and Gere (1961, p 292) indicated that the buckling forms of higher order can be obtained only by introducing certain additional constraints. For n = 3,  $p_h = 8 \frac{EI}{R^3}$  or 2.7  $p_o$ . For n = 4,  $p_h = 15 \frac{EI}{R^3}$  or 5  $p_o$ . Williamson and Huff (1961, p 42) used 15  $\frac{EI}{R^3}$  as their buckling criterion.

The hydrostatic value for the critical buckling pressure is based on the external forces remaining normal to the surface of the ring when buckling occurs. Boresi (1955, p 101) has shown that the coefficient on  $\frac{\text{EI}}{\text{R}^3}$  in equation 3.2 is 4.5 for the fundamental buckling mode if the external forces are assumed to remain directed toward the original center of the ring instead of normal to the surface. Bodner (1958) showed that the coefficient is 4 for a constant-directional-pressure force system.

The foregoing observations indicate some of the potential weaknesses in the hydrostatic assumption. A slightly different assumption in the action of the surface traction could change the critical buckling pressure by 50 percent.

Anderson and Boresi (1962) investigated a nonuniform load distribution of the form  $p = p_a \sin^2 \theta$ , Fig. 3.4, where  $p_a$  is the peak pressure at the crown. For centrally directed forces,  $p_{cr}$  (average) = 4.5  $\frac{EI}{R^3}$ , which was identical with the uniform load case where  $p_{cr}$  (average) is the total load divided by the circumference. This implies that the specific load distribution may not be overly critical in some cases.

For the test specimens of cylinder groups A, B, D, and E,  $p_0 = 135$ psi and for group C,  $p_0 = 5.1$  psi from equation 3.2 for the lowest mode.

Other investigators, e.g. Donnellan (1964), have tested cylindrical shells in which the longitudinal boundaries were supported and as a result the theoretical buckling equation became a function of the cylinder length, *i*. Timoshenko and Gere (1962, p 478) derived the expression for a simply supported shell,  $w = \frac{\partial^2 w}{\partial x^2} = 0$  where w is the deflection of the middle surface in the radial direction and x is the cylinder coordinate in the longitudinal direction, Fig. 3.1.

$$\mathbf{p}_{t} = \frac{\mathbf{E}\mathbf{h}}{\mathbf{R}(\mathbf{n}^{2}-1)\left(1+\frac{\mathbf{n}^{2}\boldsymbol{s}^{2}}{\boldsymbol{x}^{2}\mathbf{R}^{2}}\right)} + \frac{\mathbf{E}\mathbf{I}}{(1-\boldsymbol{v}^{2})\mathbf{R}^{3}}\left(\mathbf{n}^{2}-1+\frac{2\mathbf{n}^{2}-1-\boldsymbol{v}}{1+\frac{\mathbf{n}^{2}\boldsymbol{s}^{2}}{\boldsymbol{x}^{2}\mathbf{R}^{2}}}\right) \qquad 3.3$$

where  $p_t$  is the theoretical buckling pressure, and h is the wall thickness. The number of half-waves, n, into which the shell buckles increases as the length of the shell decreases and as the thickness of the shell decreases. Taking the limit of equation 3.3 as the length becomes long (approaches infinity) yields the equation for a long tube or structure

$$P_t = \frac{(n^2 - 1)}{(1 - v^2) R^3} \frac{EI}{R^3}$$

3.4

where v is Poisson's ratio of the cylinder material.

For a value of v = 0.3, equation 3.4 for a long cylindrical shell differs from equation 3.1 for a ring by only 10 percent.

Armenakas and Herrmann (1963) reanalyzed the shell case and presented convenient graphs to allow rapid assessment of the critical buckling number n corresponding to values of S/R.

3.2.2 Soil Medium Approximated by Elastic Support

Cheney (1963, p 41) derived an expression for the critical buckling pressure  $(p_c)$  of a ring with radial elastic support, Fig. 3.5.

$$p_c = (n^2 - 1) \frac{EI}{R^3} + \frac{k_z R}{n^2 - 1}$$
  $n \ge 2$  3.5

in which

$$n_{\rm cr} = \sqrt{1 + \sqrt{\frac{k_z R^4}{EI}}} \ge 2$$
 3.6

This leads to a convenient approximation

$$\mathbf{p}_{c} = 2\sqrt{k_{z}\frac{EI}{R^{2}}}$$
3.7

where  $k_z$  is the spring constant in psi per in. of radial deflection. Cheney (1964) pointed out that equation 3.7 underestimates the buckling load no more than by 10 percent for n greater than 5 and less than 1 percent for n greater than 10. For vanishing values of  $k_z$  and for n less than 5, the exact expression, equation 3.5, must be used because equation 3.7 is not suited to small values of the spring constant or n.

The great difficulty involved in applying this type of equation is the evaluation of an appropriate spring constant,  $k_z$ , for the soil. To facilitate comparison, equation 3.7 can be rewritten as

$$p_{c} = 2\sqrt{k_{z}R^{4}}\sqrt{\frac{EI}{R^{3}}}$$
3.8

Meyerhof and Baikie (1963, p 13) arrived at an elastic buckling equation by modifying the theory of flat plates on an elastic foundation. Their equation may be written as

$$P_{\rm m} = \frac{(n+1)^2 - 1}{1 - v^2} \frac{\rm EI}{\rm R^3} + \frac{(1 - v^2) \, \rm k_{\rm m} \rm R}{(n+1)^2 - 1} 3.9$$

where k is the coefficient of soil reaction ("subgrade modulus").

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For large values of n this can be reduced to

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$$p_{\rm m} = 2 \sqrt{\frac{k_{\rm m}^{\rm EI}}{(1 - v^2)R^2}}$$
 3.10

or

$$p_{m} = 2 \sqrt{\frac{k_{m}R}{(1 - v^{2})}} \sqrt{\frac{EI}{R^{3}}}$$
 3.11

Equation 3.11 differs from Cheney's equation, 3.8, only by the factor  $(1 - v^2)$ .

Luscher and Höeg (1964, p 35) used an approach of Hetényi (1946) to arrive at an equation for critical buckling pressure  $(p_{\ell})$ .

$$p_{l} = 2\left(\sqrt{\frac{k_{l}R^{3}}{EI} + 1} - 1\right)\frac{EI}{R^{3}}$$
 3.12

where

$$n_{c.r} = \sqrt[4]{\frac{k_{\ell}R^3}{EI} + 1}$$
 3.13

These can be simplified for higher order buckling modes to

$$\mathbf{p}_{\mathbf{f}} = 2\sqrt{\mathbf{k}_{\mathbf{f}}} \frac{\mathbf{EI}}{\mathbf{R}^3} = 2\sqrt{\mathbf{k}_{\mathbf{f}}} \sqrt{\frac{\mathbf{EI}}{\mathbf{R}^3}} \qquad 3.14$$

and

$$n_{cr} = \sqrt[4]{k_{\ell} \frac{R^3}{EI}} 3.15$$

where  $k_g = \text{coefficient}$  of elastic soil reaction (having the units psi per strain). Luscher and Höeg (1964, p 143) expressed  $k_g$  in terms of the constrained tangent modulus of the soil and the thickness of the soil support. For the Ottawa sand which they used, the equation was written as

$$P_{R} = 780 \left[ \frac{EI}{R^{3}} f(R) \right]^{5/6}$$
 3.16

where f(R) is a function of the depth of burial.

Newmark and Merritt (1963) considered a similar problem.

All of the above can be summarized by the following:

$$p_c = 2 \sqrt{k_z R} \sqrt{\frac{EI}{R^3}} \ge \frac{3EI}{R^3}$$
 3.8

$$p_{\rm m} = 2 \sqrt{\frac{k_{\rm m}R}{(1 - v^2)}} \sqrt{\frac{EI}{R^3}}$$
 3.11

$$\mathbf{p}_{\boldsymbol{\ell}} = 2\sqrt{k_{\boldsymbol{\ell}}} \qquad \sqrt{\frac{\mathbf{EI}}{\mathbf{R}^3}} \qquad 3.14$$

The application of this type of formula revolves around an ability to arrive at an appropriate value of the coefficient of soil reaction. This will be discussed in Chapter 6.

## 3.2.3 Soil Medium Approximated by an Elastic Medium

Forrestal and Herrmann (1964) derived a buckling equation for a long cylindrical shell subjected to uniform external pressure exerted by a surrounding elastic medium, Fig. 3.6. The solution for the unbonded case (shear stresses between the shell and the medium are absent) can be expressed as

$$P_{f} = \frac{(n^{2} - 1)}{(1 - v^{2})} \frac{EI}{R^{3}} + \frac{E_{g}}{(1 + v_{g})(1 - 2v_{g})(n + 1) + n} \qquad 3.17$$

where  $p_{f}$  is the critical buckling pressure,  $E_{g}$  is the Young's modulus of the medium, and  $v_{g}$  is the Poisson's ratio of the medium. Solutions for the bonded case were also presented but were more complicated and did not give results which varied greatly from those for the unbonded case.

### 3.3 Inelastic Action

After the cross section has yielded in hoop compression, it can

continue to yield or strain for some time before structural collapse. It is hypothesized that such failure can be defined by the judicious choice of a ductility factor. Newmark and Haltiwanger (1962) defined this factor,  $\mu$ , as the ratio of the maximum deflection to the deflection at yield. Ductility factors for compression members have been assumed to be in the range 1.3 to 1.5.

#### 3.4 Characteristic Ring Parameter

In order to compare the results of various tests run by different investigators, it is necessary to have a parameter by which the ring can be adequately described. Various groupings have been used, e.g. radius to thickness ratio, diameter to thickness ratio, and these quantities weighted in some fashion by the modulus of elasticity.

The quantity  $\frac{EI}{R^3}$  appears as a parameter in all of the aforementioned buckling equations and appears to be a convenient index for the elastic action of rings.

Stiffness can be defined as the force required to produce a unit deflection. For a large variety of loading configurations this is a function of  $\frac{\text{EI}}{\text{R}^3}$ . Fig. 3.7 illustrates a number of these loading conditions, many of which were investigated by Lane (1960, p 287). Point load, P (Fig. 3.7a):

$$\frac{P}{\Delta_{v}} = 6.7 \frac{BI}{R^3}$$
3.18

60° triangle (Fig. 3.7b):

$$\frac{p(2R)}{\Delta_v} = 29 \frac{E1}{R^3}$$
 3.19

90° triangle (Fig. 3.7c):

$$\frac{p(2R)}{\Delta_{v}} = 22 \frac{BI}{R^{3}}$$
 3.20

120° triangle (Fig. 3.7d):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 19 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.21

180° triangle (Fig. 3.7e):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 18 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.22

Parabolic (Fig. 3.7f):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 14 \frac{\mathbf{EI}}{\mathbf{R}^3}$$
 3.23

Uniform (Fig. 3.7g):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 12 \frac{\mathbf{EI}}{\mathbf{R}^3} \qquad 3.24$$

Side support (Fig. 3.7h):

$$\frac{p(2R)}{\Delta_v} = 12 (1 - q) \frac{EI}{R^3}$$
 3.25

Uniform radial (Fig. 3.)a):

$$\frac{\mathbf{p}(2\mathbf{R})}{\Delta_{\mathbf{v}}} = 2 \frac{\mathbf{E}\mathbf{h}}{\mathbf{R}} \qquad 3.26$$

where  $\Delta_{y}$  is the decrease in vertical diameter, q is the ratio of the horizontal to the vertical pressure, and h is the ring wall thickness.

It also appears that the parameter  $\frac{EI}{R^3}$  may provide a means for differentiating between so-called stiff and flexible buried cylinders. The lowa Formula (Fig. 2.1c) can be rewritten as

$$\frac{p(2R)}{\Delta_{h}} = \frac{0.061(k_{B}R) + \frac{SI}{R^{3}}}{0.083}$$
 3.27

where  $\Delta_{\rm h}$  is the increase in horizontal diameter. If a flexible structure is defined as one whose stiffness,  $\frac{\rm HI}{\rm R^3}$ , has less than a 10 percent influence on elastic deformation relative to the influence of the soil, then, from equation 3.27, a stiff structure is one in which

$$\frac{EI}{R^3} > 0.61(k_sR)$$
 3.20

In a dense sand medium (with  $k_{s}R = E' = 1000$  as suggested by Watkins and Nielson (1964, p 173)), a cylinder is stiff if  $\frac{EI}{R^3} > 610$  psi from equation 3.28. In a clay (with E' = 900), it is stiff if  $\frac{EI}{R^3} > 550$ psi. These stiffness values are greater than those required to prevent buckling for overpressures lower than 1500 psi.

Other approaches have been suggested to arrive at relative stiffness. Meyerhof and Baikie (1963) indicated that the relative stiffness, S , of a culvert with respect to the soil is

$$S = \sqrt[4]{\frac{EI}{(1 - v^2)k_m}} 3.29$$

 $\mathbf{or}$ 

$$S = \sqrt{\frac{3}{2(1 - v_s^2)EI}}{(1 - v^2)E_s}$$
 3.30

where  $v_s$  is Poisson's ratio of the soil. Davisson\* suggested that relative stiffness,  $S_1$ , could be expressed as

$$S_{1} = \sqrt[3]{\frac{EI}{RE_{s}}} \qquad 3.31$$

and that a typical discriptor, TD , would be

$$ID = \frac{R}{S_1}$$
 3.32

No numerical limits have been established to differentiate between stiff and flexible structures on the basis of these equations.

<sup>\*</sup> Private communication with M. T. Davisson, Professor, Department of Civil Engineering, University of Illinois, June 1964.

Qualitatively, a flexible structure may be thought of as one which deforms (vertical change or volumetric change) more than the medium replaced would have. However, this concept has its greatest applicability in the assessment of overall arching.

Flexibility, in the structural sense that it will deform sufficiently to mobilize the passive resistance of the side-supporting soil, appears to be assured for a structure made of ductile material whose value of  $\frac{\text{EI}}{\text{R}^3}$  is less than about 600 psi.

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#### CHAPTER 4. EXPERIMENTAL PROCEDURE

## 4.1 Description of Cylinders

### 4.1.1 Considerations in Selection of Design

A number of practical considerations were influential in the selection of the cylinder material and the geometric dimensions for the tests.

Aluminum was selected for the cylinder material because it, in general, is not strain-rate sensitive according to Steidel and Makerov (1960) and Smith (1963). It has a face-centered, cubic, crystalline, lattice structure and exhibits a continuous stress-strain curve with no sharp yielding zone. Steel was rejected because of its unpredictable yield strength under dynamic loading. Massard and Collins (1958) and Wright and Hall (1964) have proposed methods of taking this strain-rate effect into account, but it was considered best to avoid adding this parameter to the study. Plastics are made of long chain molecules which possess no ordered geometric pattern of structure, and hence are not only strain-rate sensitive but also experience a brittle failure under rapid loading as indicated by Dietz and McGarry (1956) and Hall (1958).

The relative size of the cylinders was dictated by the dimensions of the University of Illinois 2-ft-diameter, 500-psi, loading device. As a result, it can be assumed that for shallow burial no load was lost due to the effect of sidewall friction, and hence that the free-field vertical soil pressure immediately above the cylinder was equal to the surface overpressure. Measurements by Hanley (1963) have shown this to be a reasonable assumption.

The specific cross-sectional dimensions were determined by consideration of two factors. First, it was essential to have specimens that would fail under the maximum available pressure. In this regard, it was also desirable to take full advantage of the high pressure capability available by concentrating on specimens which would be too strong for ultimate strength studies in other facilities. Second, in view of the high cost of specimen preparation and the desirability of testing a large number of cylinders, commercially available tubing was sought.

The length was governed by the desire to have a somewhat realistic proportion between length and diameter, and by the need for enough length to smooth out any local disturbances caused by the presence of either the outside strain gages or end walls. Also, the length should be long enough to allow two-dimensional behavior and short enough to fit conveniently into the tank.

The closure plates (end caps) for the ends of the cylinder were designed so that no axial loading would be transferred to the cylinder, while at the same time retaining free radial motion.

4.1.2 Cylinder Material

Although all of the cylinders are made of aluminum, alloys with three different, nominal yield strengths were involved. The stress-strain properties of the materials were experimentally obtained and are discussed in Appendix A. The modulus of elasticity, E, was found to be a constant value,  $10 \times 10^6$  psi. Two yield values were determined: a lower yield point,  $\sigma_{y1}$  (which is hard to define and probably no more accurate than ±10 percent), corresponding to the first noticeable deviation from elastic behavior; and an upper yield point,  $\sigma_{y2}$ , corresponding to the

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stress which would result in 0.2 percent permanent strain. These values are summarized in Table 4.1.

4.1.3 Cylinder Geometry

The outside diameter, d , for all cylinders was 3.5 in. Micrometer measurements of the horizontal and vertical diameters prior to each test indicated that the greatest deviation was  $\pm 0.5$  percent. The larger diameter was oriented vertically for the test. The length,  $\ell$ , was a constant 10.5 in., making the length-to-diameter ratio for all cases equal to 3. Two wall thicknesses were used, 0.065 in. and 0.022 in. No deviation in thickness was found to be greater than  $\pm 0.001$  in. A longitudinal section of a cylinder is shown in Fig. 4.1, and the geometric values are summarized in Table 4.1.

4.1.4 End Conditions

The conditions at the ends of the cylinder represent a free boundary. The end caps prevented the transfer of any axial load to the cylinder and the clearance of 0.05 in. at each end was sufficient to allow for radial motion. One layer of commercial, paper masking tape was used to hold the cylinder in place between the end caps during handling and placement in the soil.

4.1.5 Natural Period of Vibration

In dynamic problems it is sometimes necessary to know the natural period of vibration of the structure for all loading conditions except a step pulse. For circular, cylindrical structures buried underground the procedure for determining the period is not well established. However, a good approximation can be made by finding the period of a cylinder in air and making appropriate corrections to account for the soil.

The natural period of the pure radial vibration of a complete

ring is given by Timoshenko and Young (1955, p 426) as

$$T_{c} = 2\pi \sqrt{\frac{\gamma R^{2}}{Eg}}$$
 4.1

where

 $T_c = natural compressive period$ 

 $\gamma$  = specific weight

R = radius of the center line of the ring

E = modulus of elasticity

g = acceleration due to gravity

For this study  $\gamma = 169 \text{ lb/ft}^3$ ,  $E = 10 \times 10^6 \text{ psi}$ ,  $g = 32.2 \text{ ft/sec}^2$ , and R = 1.72 in. (groups A, B, D, E) or 1.74 in. (group C). The calculations yield for all cylinders

$$T_{2} = 0.06 \text{ msec}$$
 4.2

For comparison, consider the period of the fundamental mode of flexural vibration given by Timoshenko and Young (1955, p 429) as

$$\Gamma_{f} = 2\pi \sqrt{\frac{5}{36}} \frac{\gamma A R^{4}}{EgI}$$
 4.3

where

Tf = natural flexural period
A = area of the cross section perpendicular to the ring
center line

I = moment of inertia of the cross section perpendicular to the ring center line

This may be rewritten as

$$T_{f} = \frac{R}{h} \sqrt{\frac{5}{3}} 2\pi \sqrt{\frac{\gamma R^{2}}{Eg}}$$
 4.4

where h = thickness of the ring.

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The substitution of equation 4.1 into equation 4.4 yields

$$T_{f} = \frac{R}{h} \sqrt{\frac{5}{3}} T_{c}$$
 4.5

For this study h = 0.065 in. (groups A, B, D, E) or 0.022 in. (group C). The calculations yield

$$T_f = 1.9 \text{ msec}, \text{ groups A}, B, D, \text{ and E}$$
 4.6  
 $T_r = 5.6 \text{ msec}, \text{ group C}$  4.7

The soil acts in two ways to modify the foregoing expressions for the natural period. It tends to stiffen, and at the same time to add mass to the structure. The effect of the mass of soil, virtual mass, which must be accelerated along with the buried structural elements can be treated in the manner suggested by Merritt and Newmark (1964, p 23); but, the deflections observed in this study for the small cylinders were of such small magnitude that it is unlikely that any appreciable amount of additional mass should be included. The stiffening effect is even less susceptible to quantitative assessment.

4.2 Description of Soil

### 4.2.1 Considerations in Selection of Test Soils

Although considerable thought is being given to what soil parameters govern soil-structure interaction, no complete answer is presently available. Therefore, it was desirable to use soils at each end of the spectrum,\* and at the same time soils whose shear strength and

<sup>\* 1</sup>st Lt. A. J. Hendron, Jr., Ph.D., "A Short Technical Note on the Extremes in Soil Types in Regard to Dynamic Soil-Structure Interaction," Vicksburg, Miss., July 22, 1964.

stress-strain properties could be documented for future reference. A new soil environment was built for every cylinder; hence, the in-place properties of the soils used had to be reproducible. Dense, dry sand and a clay of high plasticity were selected. The sand was uniformly graded because a given density was thought to be more reproducible in a uniformly graded sand than in a well-graded sand.

### 4.2.2 Sangamon River and Cook's Bayou No. 1 Sands

The Sangamon River sand has been used extensively in tests at the University of Illinois. It was used in a dense  $(D_r = 78\%)$ , dry condition as the soil environment for the testing of cylinder groups A, B, and C. The Cook's Bayou No. 1 sand  $(D_r = 79\%)$  has been used for several experiments at WES; extensive, dynamic one-dimensional and triaxial tests are planned in the near future to expand the knowledge of its properties. It was used for group E. The characteristics of both sands, together with the placement techniques employed, are outlined in Appendix B.

4.2.3 Buckshot Clay

This particular clay (CH) was selected for the group D cylinders because of the experience at WES in its use. However, even with this kind of knowledge available, great difficulty was experienced in developing placing methods adaptable to this study. The properties and placement techniques are discussed in Appendix C.

4.3 Loading Devices

Experimental work in this area has required the development of new testing machines.

4.3.1 Illinois

The equipment used in the first stage of this study was

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originally developed by Egger (1957) and later modified to permit simulation of blast loading by Sinnamon and others (1961). Its capabilities are described by Sinnamon and Newmark (1961), and it has recently been used by Hanley (1963) to study the interaction between sand and vertically oriented cylinders.

The container is a vertical cylinder 26-3/4 in. high and 23-1/4in. in diameter. A 1/32-in.-thick neoprene diaphragm is placed over the soil surface to prevent gas penetration. Then a spacer ring is positioned, followed by the static or dynamic loading head. The device is illustrated in Fig. 4.2. Both the static and dynamic loads are provided by a compressed gas system. Although the equipment is capable of producing rise times in the neighborhood of 3 msec by using helium gas, this study was conducted with nitrogen gas because it is less expensive and because the 3 msec rise time apparently offered little advantage over the 13 msec rise time (rapid) with nitrogen gas. A typical overpressure-time relation is shown in Fig. 4.3. No reflection of the incident wave on the bottom was noted.

4.3.2 WES

Cylinder groups D and E were tested in the Small Blast Load Generator (SBLG) facility at WES. This was the first extensive experimental program completed in the SBLG and hence a number of problems in technique had to be resolved during the course of the investigation. The dynamic overpressure is applied by the detonation of two parallel lines of PETN in the form of primacord. The effective overpressure-time relation (dynamic) is shown in Fig. 4.3. The early part of the curve was obtained by averaging the maximum and minimum points in adjacent oscillations.

Although the amplitude of the oscillations varied as much as  $\pm 50$  percent from the average, the impulse was so small (10,000 and 20,000 cps ringing) that the approximation in Fig. 4.3 is justified. The high-frequency signals were probably caused by the nonshock isolated gage mounts. The pressure distribution on the surface is within  $\pm 10$  percent of being a plane wave according to Kennedy and Sadler (1965).

The loader is a cylindrical ring device, 46-3/4 in. in diameter. For these tests an average soil replacement depth of 2 ft was used. The layout is shown in Fig. 4.4. The static tests of group D were run with a rigid concrete base (III). The static tests of group E along with the dynamic tests of both D and E were conducted with a pseudo-infinite base (II) to avoid the dynamic disadvantages of the rigid base.

The "infinite" base is a column of sand extending 9 ft below the floor level. This column had been previously loaded many times to 500 psi, and no further compaction was observed. Two feet of sand above floor level was replaced for each sand test. For the dynamic clay tests, a rubber diaphragm was inserted at floor level to separate the lower sand column from the upper 2 ft of clay.

The operation of the loading device has been outlined by Boynton Associates (1960), and the U. S. Army Engineer Waterways Experiment Station (1963) and an evaluation study is being made by Kennedy and Sadler (1965).

# 4.4 Instrumentation

### 4.4.1 General

Letal film strain gages were used to measure hoop strain on the inside and outside of the cylinders (Fig. 4.1). Static deflection gages were made from brass shim stock and individually calibrated. The transducers and techniques are discussed in Appendix D.

4.4.2 Illinois

The instrumentation used is pictured in Fig. 4.5. The active strain gage on the cylinder was one arm of a four-arm bridge. The dummy gages were mounted on isolated metal strips outside the test tank. Multiconductor cable was used initially, but it was found that two-conductor shielded cable provided a better barrier to spurious noise in the system.

The eight hoop strain gages were hooked to a bank of Consolidated Electrodynamics Corporation (CEC) carrier amplifiers, Type 1-127. A 12-channel CEC, direct-write, recording oscillograph Type 5-124 with available paper speeds of 0.5, 2, 8, 32, and 128 in./sec was used. The two deflection gages each formed two arms of a bridge and were fed through DANA d-c amplifiers to the oscillograph. For the static tests, the slowest paper speed was used. A timing trace of 2 cps and one reference (dead) trace completely utilized all of the available channels. The overpressure was read on an auxiliary Bourdon gage with the timing trace interrupted at predesignated pressure levels. Modifications were made for the rapid tests. The output of the strain gage amplifiers was split so that it was placed on both the oscillograph and a Honeywell 8100 tape recorder (as a back-up record). Additional DANA amplifiers were used to drive the tapes. The time base frequency was increased to 500 cps. The output of a Kistler Instrument Corporation, piezoelectric, pressure transducer, which was in series with a Kistler calibrator and charge amplifier, was used to record pressure. The recording paper was driven at the fastest speed possible, 128 in./sec.

The frequency response of the oscillograph system was limited to

that of the CEC 7-364 galvenometers, 500 cps. The tape system had a frequency response of at least 3000 cps and a few records reproduced directly from the tape indicated that no frequencies higher than 500 cps were present.

## 4.4.3 WES

The equipment used for group E (the first test series at WES) and the evaluation of the overpressure-time signature is shown in Fig. 4.6. The Wheatstone bridge was set up as in the Illinois tests. The Sensor Analog Module (SAM) amplifiers used are d-c, and hence the dynamic frequency response was again limited by the galvanometer capabilities, 2500 cps (CEC 7-362).

After the group E tests were completed, the SBLG facility instrumentation was moved to a separate area. The layout used for the group D tests is shown in Fig. 4.7. In this case, DANA amplifiers coupled with galvo drivers were used.

Overpressure was monitored by a pair of 1000-psi Norwood pressure transdulers, Model 211C. Additional pressure transducers were used and their output recorded on tape to gain higher frequency response (20,000 cps) in order to describe adequately the high-frequency characteristics of the pressure-time signature.

4.4.4 Sources of Error

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Fotential sources of error are present throughout the system: (1) inexact strain gage placement  $(\pm 2 \$)$ ; (2) variation in gage factor and resistance  $(\pm 1 \$)$ ; (3) amplifier nonlinearity  $(\pm 2 \$)$ : (4) galvanometer nonlinearity  $(\pm 1 \$)$ ; and (5) properties of the pressure transducers  $(\pm 5 \$)$ . These imply a confidence limit of no better than  $\pm 11$  percent in the instrumentation system.

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CHAPTER 5. ESENTATION OF EXPERIMENTAL RESULTS

## 5.1 Method of Presentation

# 5.1.1 Cylinder Coding

Table 5.1 outlines the overall testing program for the 46 specimens and identifies each cylinder with its respective soil environment, depth of burial, and type of loading. The notation used, e.g. A-3, to identify each cylinder (and thus each test) has general meaning. The alphabetic term, A, was used to identify the original 12-ft tube from which the test cylinder was cut and can be related to the stress-strain curves of Appendix A. Cylinders with a numerical designation 1 through 5 were tested statically, while those designated 6 through 10 were tested either rapidly or dynamically. In Tables 5.2 through 5.11 the tests are presented by group (A, B, C, D, E), static first, in the order of increasing depth of burial within the group.

### 5.1.2 Tables of Data

The digitized strain values were taken from the oscillograph records at points corresponding to specific values of the overpressure to obtain a cause-and-effect relation. In the dynamic tests, peak strain values were recorded. These experimental strain values, together with diameterchange values (for static tests only), are listed in Tables 5.2 to 5.11 with respect to overpressure.

Use of a dash instead of a number indicates that the results were lost due to instrumentation difficulties. The values of stress, threat, and moment are also listed in the tables. The gage locations are identified in Fig. 4.1.

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## 5.1.3 Data Plots

The values of strain were, in general, not plotted directly in Figs. 5.1 to 5.43 because an appropriate scale to show the large inelastic values would have masked the much smaller elastic strains. The stress to cause yield and the thrust to cause yield are shown by horizontal dotted lines in each figure. "First yield"  $(\sigma_{y1})$  represents the stress at a point where the slope of the stress-strain curve departs from the initial elastic slope (E). The yield value corresponding to 0.2 percent permanent strain is the "0.2 percent offset yield"  $(\sigma_{y2})$ . The diagonal dotted line labeled "uniform radial load" represents the theoretical relation derived for a uniform radial load equivalent in magnitude to the overpressure, Fig. 2.1d.

Stress, thrust, moment, and diameter change (static tests only) are plotted as ordinates with respect to the surface overpressure as the abscissa.

The symbols used to identify a gage location are presented on each figure and are consistent throughout. The inside gages are represented by open symbols and the outside gages by closed symbols. The cross sections are identified by the applicable open symbol.

#### 5.2 Computations

#### 5.2.1 Moment and Thrust Computation

The moment and thrust at a cross section were calculated from



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where  $M_y$  is the moment in the y or tangential direction, Fig. 3.1, in units of pounds (inch-pounds per inch), and  $N_y$  is the thrust or in-plane force in the tangential direction in units of pounds per inch. For the elastic case these can be reduced to

$$M_{y} = (\epsilon_{e} - \epsilon_{i}) \frac{Eh^{2}}{12}$$
 5.3

$$N_{y} = (\epsilon_{e} + \epsilon_{i}) \frac{Eh}{2} \qquad 5.4$$

where  $\epsilon_e$  is the exterior strain, and  $\epsilon_i$  is the interior strain at the cross section in inches per inch. Compressive strains and thrust are considered positive in the presentation. Moment tending to compress the external fibers is positive.

## 5.2.2 Computer Program

To reduce the large mass of strain data to applicable stress, thrust, and moment values, a program (13-G1-Z5010) was written in FORTRAN for the WES, GE 225 computer. The aluminum stress-strain curves of Appendix A were input in a discrete number of linear segments and a "table lookup was utilized to compute the elastic and inelastic stress. The strain distribution was assumed to be linear across the section, Singer (1951, p 409), so that the expressions for moment and thrust, equations 5.1 and 5.2, could be numerically integrated for the nonelastic case. The program assumes that the material stress-strain properties are the same in both tension and compression and that any unloading takes place along the original load curve.

## 5.2.3 Computation of q

Values of q are listed in Tables 5.2 to 5.11. As used in this

context, q is not a coefficient of earth pressure, but merely defines the atio of the average thrust at the crown and invert divided by the average thrust at the spring line. Values of q are plotted in Figs. 6.1 to 6.3. 5.3 Mode of Failure

All of the cylinders that failed, failed by a catastrophic snapthrough (caving) of the crown. A noise was heard at the moment of failure and all of the strain gage traces were instantaneously driven off the oscillograph, either by being overranged or by shorting out electrically. The last recorded strains in the tables are those at the moment of failure.

The failed cylinders are shown in Figs. 5.44 and 5.45. The distorted cross section of two cylinders which did not fail are shown in Fig. 5.46 (the strain gage wires are evident in D-6), and the postfailure clay configuration is illustrated in Fig. 5.47. A plot of overpressure at failure versus depth of burial is shown in Fig. 5.48.

5.4 Stress, Moment, and Thrust

The cylinder groups are presented in the order A, B, C, E, and D because the first four groups were in a sand medium and the last in clay.

5.4.1 A Group

The static test data are presented in Table 5.2 and plotted in Figs. 5.1 through 5.6. An air line broke at 400 psi during test A-3. Fig. 5.4, test A-3A, presents the data up to that point. The line was repaired, the gages were rezeroed, and a second test, A-3B, Fig. 5.5, was run up to 500 psi. The values of stress, thrust, and moment listed for test A-3B were computed by the computer program on the assumption of no residual strain. Sample calculations based on the more realistic assumption of residual strains from test A-3A indicated that the listed values are no more than about 10 percent low.

The deflection gages were not suitable for the rapid testing, and hence data from them do not appear in Table 5.3 nor in Figs. 5.7 through 5.11.

5.4.2 B Group

The static test data are presented in Table 5.4 and plotted in Figs. 5.12 through 5.17. The B group was the first group to be tested, and B-1 was the first cylinder. Test B-1A, Fig. 5.12, terminated at 300 psi because no higher pressure was attainable with the loading device. A subsequent modification in the O-ring configuration allowed the device to attain its 500-psi static capacity. Test B-1 was rerun, test B-1B, Fig. 5.13, and the cylinder failed at 315-psi overpressure.

The rapid test data are presented in Table 5.5 and plotted in Figs. 5.18 through 5.22.

5.4.3 C Group

The static test data are presented in Table 5.6 and plotted in Figs. 5.23 through 5.27. The rapid test data are presented in Table 5.7 and plotted in Figs. 5.28 through 5.32.

5.4.4 E Group

The static tests were run as duplicates to check the tests of the A group. Test data are presented in Table 5.8 and plotted in Figs. 5.33 through 5.35. The dynamic results (peak strain values) are presented in Table 5.9 and plotted in Figs. 5.36 and 5.37. The initial pressure rise of the dynamic pressure wave, Fig. 4.3, approximates a step pulse. For this region a strain-pressure relation is unmanageable. Therefore, the dynamic results are plotted with respect to the circular angle  $\theta$  (Fig. 4.1) for the various overpressures attained. No failures resulted from the maximum available, nominal overpressure of 250 psi.

5.4.5 D Group (Clay)

The static test data are presented in Table 5.10 and plotted in Figs. 5.38 through 5.42. The dynamic results are presented in Table 5.11 and plotted with respect to the circular angle  $\theta$  in Fig. 5.43. The values of stress, thrust, and moment were computed by the computer program on the assumption of no initial strain. Sample calculations, which took into account the strains impressed during placement, indicated that the values listed in Tables 5.10 and 5.11 are no more than about 10 percent low.

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### CHAPTER 6. ANALYSIS AND INTERPRETATION OF TEST RESULTS

Initially this discussion will concern Figs. 5.1 through 5.48; then other detailed comparisons of pertinent aspects of the data will be treated.

6.1 Overall Structural Response

6.1.1 A Group (Sangamon Sand)

Fig. 5.1, test A-1 (Z = 0 in.), depicts the structural response of a relatively stiff cylinder as it progresses toward failure under static loading. This is a typical case only for cylinders buried at depths approaching zero depth of burial. It is evident that the stress curves are not linear functions of the applied pressure even in the elastic range of the cylinder material; the lower stresses (those tending to tension) are the ones most susceptible to nonlinear behavior. The agreement of the stress levels for gages 2 and 4, and 2a and 4a indicates that the cylinder experienced generally symmetric response about the vertical axis. The crown and invert at this very shallow burial did not exhibit this agreement in response. The stress at many gage points was greater than the first yield stress of the cylinder material. Only the stress recorded for the outside gage at the crown, la , tended to pass the 0.2 percent offset yield stress of the material (at incipient failure).

Thrust is a more nearly linear function of overpressure than the stress at any gage point. The thrusts at the four cross sections are nearly equal below 150 psi; but at high pressures the thrust at the invert is considerably lower than the thrust at the crown or spring line for the case of shallow burial.

The decidedly nonlinear variation of moment with overpressure above 100 psi is the consequence of the cylinder readjusting itself under load and probably of the load distribution changing. It is important to note how the magnitude of the spring line moment decreases for input pressure greater than 200 psi. It is at this pressure that the stresses exceeded the first yield stress of the structural material. The change in sign of the crown moment is of concern. For the structure to assume an elliptical shape (with the major axis horizontal), it would seem that the crown moment would have to be positive throughout the loading. However, this is not the case for pressure levels below 210 psi. Coupled with this, the diameter changes are extremely small for the first 210 psi of loading. This type of reversal of curvature at the crown was not an isolated occurrence. It is shown in the results of test A-5 in Fig. 5.2 and in other cylinders which are very close to the surface boundary and susceptible to collapse. There are a number of possible explanations for this phenomenon.

- The vertical axis was slightly greater than the horizontal axis, and this by itself may have influenced the sign of the moment prior to incipient failure. However, if this were significant it would have influenced the moments at deeper depths of burial.
- 2. The external strain gages and their respective protective covering could cause load concentrations away from the gage locations by activating local arching. But, this would not be the case with the depth of burial, Z, equal to zero.

- 3. Nonuniformity in the soil medium could cause uneven stress distribution. Again, this should be a random occurrence, while the phenomenon is systematic.
- 4. The tendency to buckle in a mode other than the lowest mode could cause local moment anomalies. Higher order buckling modes would have node points occurring in a random fashion even though collapse came by a full snap-through (caving) of the crown. But, here too the occurrence would be random.
- 5. The proximity of the crown to the surface boundary at very shallow burial, relative to the proximity of other points, is much more significant than at deeper depths of burial. The load at the crown is fixed, but local arching could have caused an uneven load distribution. At the deeper depths enough soil would be present to smooth out the local variations.

DaDeppo (1963, p 30) concluded that the magnitude of initial deformation in arches was important in controlling the flexural response. He was most concerned with variation in the initial shape induced by backfilling. However, the conclusion would apply regardless of how the variations in initial shape came about. Random deviations of the cylinder from circularity could result in random moment response. But, the moment response in the present investigation was systematic and repeatable.

Robinson (1964) recorded moments on a cylinder at every 45-degree point, and they were all of the same sign. He felt that this was due to local arching of the soil at the contact between the external strain gages

and the soil. However, the data were not reproducible.

It is the writer's opinion that the most plausible explanation of the negative moment is directly related to the proximity of the surface boundary causing local arching to neighboring elements of the cylinder. The buildup in pressure and subsequent nonuniform loading become less significant at the higher pressures. At depths greater than 1/4d (d/4) the crown moment is positive, Fig. 6.1. This indicates that the crown response is greatly influenced by the surface boundary at depths shallower than d/4. Overall arching can be applied to the crown at depth, but not at very shallow burial.

Test A-5 (Z = 3/16 in.), Fig. 5.2, agrees very well with test A-1 (Z = 0 in.), Fig. 5.1, both qualitatively and quantitatively, with two exceptions. First, the overpressure required to cause failure is higher for A-5. Second, the invert moment is negative in A-5 and positive in A-1. Again, for the elliptical geometry one would expect this moment always to be positive. However, it appears to be positive or negative in a random variation. This could be a result of geometric imperfections, incipient high buckling modes, or the character and nonuniformity of the soil bedding. The latter, noruniformity of the soil bedding, appears to be the most reasonable explanation at pressure levels below 300 psi. In many tests, A-5 (Z = 3/16 in.), A-2 (Z = 7/16 in.), etc., the moment at the invert changed from negative to positive at pressures gre .er than 300 psi. The significance of the initial bedding decreases as the pressure level increases. An exception is test A-4 (Z = 1-3/4 in.).

Also in test A-5 (Z = 3/16 in.) a vertical diameter increase was recorded at 50 and 100 psi. This is compatible with both the crown and

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invert moments being negative at that pressure.

Donnellan (1964, p 29) recorded an outward displacement of the radius at the invert of one of his shallow-buried cylinders. The present study recorded only diameter changes, and it is not possible to tell if half a diameter (the radius) increased while the other half decreased.

Test A-2 (Z = 7/16 in.), Fig. 5.3, follows the trends observed at the shallower depths except that no failure was experienced at the maximum machine loading capability of 500 psi. Additionally, the large positive bending moment at the crown observed just prior to failure in tests A-1 (Z = 0 in.) and A-5 (Z = 3/16 in.) was not encountered in this test. Also, the rate of change of moment with pressure decreased, indicating local arching.

Again at about 200 psi the rate of vertical diameter change begins to appear more rapid than below 200 psi. This is probably a result of the cylinder material reaching its yield value at several locations. The moments continue to decrease at overpressures above 200 psi.

Test A-3A (Z = 7/8 in.), Fig. 5.4, exhibits virtually identical thrust values at all four cross sections at pressure below 150 psi. However, at higher levels it establishes the generally observed trend of the spring line having the highest thrust, followed by the crown, with the invert experiencing the least amount of thrust. This is probably a consequence of the bedding providing a soil environment different from that around the crown.

The test (A-3A) was aborted at 400 psi by a broken gas line. The pressure went to zero, the line was repaired, the gages were rezeroed on the oscillograph, and a second test, A-3B, was run without touching the cylinder or the soil. From Fig. 5.5 it can be seen that some aspects of the structural response changed as much as 100 percent as a result of this

cycling of the load. This gives a graphic illustration of how initial, geometric deformations (plastic set in this case) can affect moments. The crown moment is much larger on the second cycle, and the invert moment has changed character greatly.

Test A-4 (Z = 1-3/4 in.), Fig. 5.6, underwent similar response to that of test A-3A (Z = 7/8 in.) with the exception of the invert moment which continued to remain large throughout the test.

The only variable changed between the static tests, A-1 through A-5, and the rapid tests ( msec rise time to 500 psi), A-6 through A-10, Figs. 5.7 through 5.11, was the rise time. The rapid tests in general verified the static tests, but several differences can be seen. First, the pressure necessary to cause collapse was somewhat higher in the rapid tests. This may have been due to a true increase in capacity or to the possibility that some creep mechanism was involved which resulted in failure appearing at a slightly higher pressure in the rapid tests. Second, the values of the thrust are about 20 percent higher in the rapid tests. This may have been due to inertial effects in the soil adding load to the structure. Third, the crown moment is initially positive up to about 100 psi in all rapid tests. For very shallow burial, the moment changed sign and was negative to about 250 psi; then it became positive again. Apparently, the pressure wave struck and depressed the crown, causing the initial positive moment. This occurred at about 3 msec which was slightly greater than the natural period of vibration in the first flexural mode, equation 4.6. This, of course, is much later than would be expected if equation 4.6 were directly applicable.

Although the symmetry around the vertical axis was good, test A-9

(Z = 3/16 in.), Fig. 5.8, illustrates how the spring-line moments can differ by as much as 100 percent (at 150 psi) while the spring-line thrusts agree well. Also, it can be seen that the disparity is not constant during the whole loading cycle, but rather tends to decrease as the cylinder material yields. Also, the moment changes produce deformations which tend to reduce disparities. Test A-7 (Z = 7/8 in.), Fig. 5.10, is a good illustration of the general response.

It is of interest to plot various responses of the group together, as shown in Fig. 6.1. The average spring-line thrust was calculated (refer to Tables 5.2 and 5.3) and the results of all ten tests plotted. It can be seen that all of the test results fall close together and exhibit a linear increase with respect to pressure, and that the rapid test results lie slightly higher (for a given pressure level) than the static results. Data from those cylinders which failed fall right along with those from cylinders which did not fail, indicating that thrust by itself (without some link with depth of burial) will not be an adequate failure criterion for very shallow depths of burial.

The crown moment plot shows how closely the rapid and static tests agree at pressures above 100 psi. The crown moments are always positive at depths greater than d/4.

The average of the crown and invert thrusts was divided by the average spring-line thrust to form the ratio q. This is plotted in Fig. 6.1. After experiencing a large range in values at pressures be'ow 200 rsi, the ratio settles into a band between 0.6 and 0.8. The values are least accurate in the lower pressure regions and are most influenced by the initial conditions created by the soil placement. Disregarding the few very high values,

the trend is to start at about 0.4 (which is approximately equal to the coefficient of earth pressure at rest), increase to about 1.0 as the cylinder began to deform, and then decrease slightly and become relatively constant.

The vertical diameter changes in the static tests are also plotted together. There is a decrease in diameter change with depth of burial for a given overpressure that is noticeable at pressure levels above 250 psi. This reflects the stiffening effect of the soil as the depth of burial increases.

6.1.2 <u>B Group (Sangamon Sand)</u>

The B group differs from the A group only in the value of the yield stresses. The B group had about twice the yield value of the A group.

The pressure causing failure was consistently higher in the B group, Table 5.1, indicating that the yield stress probably had some influence on the collapse pressure. However, this influence does not appear to be large in these tests.

In tests B-1A and B-1B (Z = 0 in.), Figs. 5.12 and 5.13, the effect of cycling is again seen in the character and magnitude of the crown moment. It is also significant that the effect of the cycling is not very pronounced at other locations (which did not yield during first loading). Other studies, Dorris and Albritton (1965) and Albritton and others (1965), have also shown that cycling may not affect the reproducibility more than about 20 percent as long as the cylinder material remains elastic.

Test B-3 (Z = 1-3/4 in.), Fig. 5.16, and test B-4 (Z = 2-5/8 in.), Fig. 5.17, again show that the results are reproducible. They also indicate that moment increases at a decreasing rate (but remains large until the material begins to yield).
The rapid tests, Figs. 5.18 through 5.22, yielded much the same information as the static tests. Tests B-9 (Z = 1-3/4 in.) and B-10 (Z = 2-5/8 in ), Figs. 5.21 and 5.22, illustrate the smoothing out of response that can be expected with deeper depths of burial.

A summary of the B group response is plotted in Fig. 6.2. As with the A group, the spring-line thrust is generally linear with pressure up to a level equivalent to first yielding of the material. The values of rapid test thrusts are larger than those for the static case. The vertical diameter changes fall into a pattern with each other and are lower than those of the A group, Fig. 6.1, at pressures greater than 200 psi. The q values settle into a band between 0.5 and 0.8 for pressures greater than 300 psi.

## 6.1.3 C Group (Sangamon Sand)

The C group of cylinders was only one-twentieth (1/20) as stiff as the A and B groups. The yield stress was high enough that all of the cylinder strains recorded were below the level corresponding to 0.2 percent permanent strain. The pressures required to induce failure were lower than in the A and B groups by a factor of 2 or 3. But, again, at depths greater than one-eighth the diameter no failures occurred. The moments in the C group were substantially smaller, and the moment scale for plotting was changed by an order of magnitude from that used for the B group.

Test C-l (Z = 0 in.), Fig. 5.23, experienced negative moments at all four cross sections and the vertical diameter increased at pressures above 25 psi. This was probably caused by the propensity for collapse in a high-order buckling mode.

The variability in moment response is even more evident in these

very flexible cylinders at shallow burial. Tests C-4 (Z = 3/16 in Fig. 5.24, and C-2 (Z = 7/16 in.), Fig. 5.26, both experienced positive moments at the spring line and the horizontal diameter decreased in C-4. Donnellan (1964, p 26) also recorded inward movement at the spring line of some flexible cylinders. This may be another manifestation of a tendency toward a high-order buckling mode.

The crown thrust was larger than that at the spring line in most of the C group tests. But, q was still less than 1.0 in most cases, Fig. 6.3. The invert thrust was low and probably reflects a decrease in vertical pressure between the crown and invert. This also shows up in a lower arching ratio, Section 6.4.

Rapid tests C-6 through C-9, Figs. 5.28 through 5.31, exhibited the same type curvature changes at shallow burial as the A and B groups. The initial peak positive moment occurred at about 3.5 msec which is about half the natural flexural period given by equation 4.7. Test C-10 (Z = 7/8 in.), Fig. 5.32, is a good example to validate the argument for application of the ring compression theory to flexible cylinders which are not affected by the surface boundary.

Test C-9 (Z = 7/16 in.), Fig. 5.31, exhibited the largest applied pressure, 550 psi, enco.ntered during this investigation. This was the only test in which the maximum pressure deviated from 500 psi. The response ended as usual when the pressure peaked, but the cylinder collapsed about a minute later as the pressure was about to be manually decayed. A stability problem is, of course, very sensitive to slight disturbance, but this also points to a possible creep effect reducing the resistance to buckling. The average spring-line thrust values, Fig. 6.3, show more

scatter than the previous two groups, but the exclusion of test C-2 (Z = 7/16 in.) reduces the spread considerably. Although no characteristics of the test indicated a difference, the results are not in line with the rest of the C group.

The values for the rapid tests are higher than those for the static. The q values for pressures greater than 300 psi lie in a band between about 0.7 and 1.0 with the exception of test C-2. In this test the q values are higher because the spring-line thrusts were lower than the rest of the C group.

## 6.1.4 E Group (Cook's Bayou Sand)

The cylinders used in the E group were identical with those of the A group except that they were cut from a different tube (same nominal material) and hence had a slightly different yield (Appendix A). The three static tests were run as a verification of the reproducibility of the A group results and for comparison with dynamic tests E-4, E-5, and E-6.

The thrust, moment, and diameter change results of E-3 (Z = 0 in.) are plotted together with companion values from test A-1 in Fig. 5.33. The values for thrust are comparable, but the spring-line thrusts of the E group are higher than those of the A group. The diameter change values also are higher and only the spring-line moments are compatible. E-3 failed at 205 psi, whereas A-1 failed at 270 psi. This is reasonably good agreement for such a buckling failure, but the thrust and diameter change trends suggest that the response was more unfavorable in the E test. Different mands were used in the two tests but they have about the same strength and deformation characteristics (Appendix B). If anything, the Cook's Bayou send (E group) is slightly stiffer than the Sangamon send

(A, B, and C groups). As a result, it is felt that the variation in response is a function of the two different methods of placing the sand around the cylinders. The Sangamon River sand was vibrated and rodded in, whereas the Cook's Bayou sand was sprinkled into place. This illustrates one of the difficulties inherent in comparing results from tests in which different placement techniques were used. Conservative conclusions must be drawn.

Test E-2 (Z = 7/16 in.), Fig. 5.34, exhibits the same trends as E-1, and the similarity of the thrust with A-2 is evident. Also, at pressures above 300 psi the moments show closer agreement. It is interesting to note again how the large moments tend to decrease as the cylinder material yields and loading progresses.

Test E-1 (Z = 7/8 in.), Fig. 5.35, exhibits even better agreement with its A group counterpart. However, the large crown moment at pressures below 250 psi and the greater diameter changes of the E group indicate that sprinkling placement of the sand gave a lower density and less restraint.

The recorded values of peak strain on the intrados and extrados for E-5 (Z = 7/16 in.) and E-4 (Z = 7/8 in.), Fig. 5.36, are compared with the values recorded for the static tests at the same 250-psi level (maximum dynamic pressure available). A large amount of ductility is evident in the dynamic tests. Using the analysis outlined by Newmark and Haltiwanger (1962) for a step pulse input of 250 psi and an equivalent elastoplastic resistance function for the cylinders, a theoretical ductility factor of 7 and a theoretical maximum strain of 5100 µin./in. were calculated. This theoretical strain agrees well with the observed strains which ranged between 5000 and 6000 µin./in.

The moment and thrust values are shown in Fig. 5.37. The peak thrusts are uniform around the cylinders for all three dynamic tests at the 250-psi pressure level used. The thrust values for the static and rapid tests are also very consistent with each other, whereas the moment values are widely scattered at the crown and invert.

6.1.5 D Group (Buckshot Clay)

The D group cylinders were buried in clay, but were identical with those of the A and E groups in material and geometry with the exception of a slight change in yield points (Appendix A) resulting from use of different tubes.

The static tests, Figs. 5.38 through 5.42, indicate higher bending moments and larger diameter changes than occurred in sand. The thrust values follow about the same trend as in sand. Generally, symmetric response was recorded and hence opposite gages acted as a check on each other.

The thrusts recorded in several tests, e.g., D-4 (Z = 1-3/4 in.) and D-5 (Z = 2-5/8 in.), were higher at the 45-degree cross section than at the spring line. The instability may very well be concentrated between this level and the crown.

The moments are a highly nonlinear function of overpressure and tend to decrease as the material yielded at high pressure levels, Fig. 5.41.

Ultimate-strength dynamic testing with the WES type Heaviside input is essentially a "go-no go" process. The true failure pressure can only be bracketed between a known collapse and a known survival. A tight bracket would require many tests and be extremely expensive. At the same time it would not be truly reliable because of the inherent scatter in stability problems.

The experience with sand indicated that the rapid and dynamic failure pressures would be relatively close to the static values. This proved to be the case also in clay, and the static failure pressures served as the basis for estimating required dynamic overpressures. The overpressures obtained were not always close to those requested because of variabilities in the loading apparatus. However, a reasonable bracket was obtained for two representative depths of burial, 7/8 in. and 1-3/4 in.

The results obtained from those cylinders which survived are plotted in Fig. 5.43. Results from those cylinders which failed are also plotted to shed more light on what occurred. However, these data should be considered only as guides. They were obtained from the records at incipient failure. This was extremely hard to define for the dynamic tests in which the cylinders failed.

Some instrumentation difficulties were encountered and the data from half the strain gages, Table 5.11, in test D-10 (Z = 7/8 in.) were lost because an oscillograph malfunctioned. However, the thrust values of D-8 (Z = 7/8 in.) and D-6 (Z = 1-3/4 in.) are relatively uniform. The peak moments are at the crown and are positive in sign. The permanent deformations in D-6 and D-10 can be seen from the end views of Fig. 5.46. The strains far exceeded yield in most cases, both in tension and compression, and resulted in high bending moments.

## 6.2 Diameter Change

The diameter changes were small for all tests. In order to verify the validity of the diameter change gages, the cylinder diameters were measured to the nearest one-thousandth of an inch with outside micrometers, both before and after the test (when possible). These

results are plotted in Fig. 6.4 along with the peak diameter change indicated by the diameter change gages. Reasonable verification is evident.

A vertical Collins gage was included in test E-5, and its peak output substantiates the trends.

Several observations can be made based on Fig. 6.4. The horizontal deflection stiffness,  $P_{so}/\Delta_h$ , appears to be independent of the buckling stiffness,  $\frac{EI}{R^3}$ ; but, it varies a great deal with the soil environment. The Sangamon River and Cook's Bayou sands differ by a factor of 2 for horisontal stiffness. The clay is less stiff by an order of magnitude.

Using these empirical values for horizontal stiffness, it is possible to calculate subgrade moduli from the Iowa Formula,

$$\Delta_{h} = \frac{0.166 \ p_{g} R^{h}}{EI + 0.061 k_{g} R^{h}}$$

vhere

 $\Delta_h$  = horisontal diameter increase, in.  $P_a$  = vertical pressure on top of the cylinder, psi R = cylinder radius, in.

E = modulus of elasticity of the cylinder, pei

I = moment of inertia of the cylinder cross section, in.

 $k_{a}$  = modulus of passive resistance of the soil, 1b/in.<sup>3</sup>

This can be solved for  $k_R$ , E', in terms of the other parameters where E' is called the modulus of soil reaction.

$$\mathbf{E}' = \frac{1}{0.061} \left[ (0.166 \text{ R}) \frac{\mathbf{p}_{a}}{\Delta_{h}} - \frac{\mathbf{g}_{I}}{\mathbf{R}^{3}} \right]$$

Substituting R = 1.75 yields

6.1

$$E' = \frac{1}{0.061} \left( 0.2905 \frac{p_a}{\Delta_h} - \frac{EI}{R^3} \right)$$
 6.2

Using the average values of  $\frac{P_{so}}{\Delta_h}$  calculated from the results plotted in Fig. 6.4 as  $\frac{p_a}{\Delta_h}$ , values of E' can be calculated. This assumes no change with depth of burial and is essentially true within the scatter for the range of shallow depths investigated. A trend of increasing stiffness with depth is true of the vertical stiffness. A typical calculation follows.

E' for the A group = 
$$\frac{1}{0.061} \left[ 0.2905(32,900) - 45 \right]$$
  
=  $\frac{1}{0.061} \left( 9550 - 45 \right) = \frac{9505}{0.061} = 155,900 \text{ psi}$  6.3

 $k_{g}$  for the A group =  $\frac{E'}{R} = \frac{155,900}{1.75} = 89,100 \text{ lb/in.}^{3}$  6.4

Also, from equation 6.2 one can compute

These calculations verify how little influence the buckling stiffness of the cylinders has on the deformations in competent soils such as these, under the assumptions of this mathematical model. The deformations are controlled by the stiffness of the soil. For example, in the

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computations for equation 6.3 the cylinder buckling stiffness,  $\frac{EI}{2}$ , is a negligible term relative to the horizontal soil stiffness  $\frac{P_a}{\Lambda_a}$ .

The calculated soil parameter,  $k_s R$ , is of the same order of magnitude as the moduli from the one-dimensional consolidation and triaxial bests at roughly the same pressures (Appendixes B and C).

Up to this point everything has been analyzed in terms of the overpressure,  $P_{so}$ , on the surface. Here it was assumed that the pressure,  $p_a$ , at the level of the cylinder crown was equal to the surface pressure. This is true by definition only when the cylinder is at zero depth of burial. However, the assumption is satisfactory within the limits discussed in Section 6.3.

# 6.3 Arching Ratio

Overall arching may be assessed by summing forces in the vertical direction above the cylinder. The thrust at the spring line represents a vertical force as does the surface pressure integrated over the area. The arching ratio, AR , is defined as the average spring-line thrust divided by the overpressure times the radius.

$$AR = \frac{\frac{N_{y}(avg)}{P_{gR}}}{P_{gR}}$$
6.5

These ratios have been calculated from the results of the static tests and are plotted in Fig. 6.5.

The A and B groups verified one another well below 200 psi. At that pressure level the A group cylinders began to yield, the moments began to decrease, and hence the cylinders stiffened as a result of approaching more closely a compression mode. The arching ratio increased until such time as the whole cross section yielded, at about 300 psi. After that, the arching ratio decreased.

It appears that the B group began to stiffen at 450 to 500 psi. The moments decreased and the arching ratio began to increase. If the trend were to continue at higher pressure, it would be compatible with the A group behavior.

The E group began with a higher arching ratio than the A group, but at pressures above 250 psi they are similar. These groups had the same buckling stiffness,  $\frac{\text{EI}}{R^3} = 45$ , but as has been pointed out the soil placement techniques differed. This indicates that initial soil differences (densities in the immediate vicinity of the cylinder, Appendix B) created by placement techniques may not be important after the soilstructure system has readjusted under 200-psi overpressure.

It is the writer's opinion that it is appropriate to express cylinder response in terms of the pressure,  $p_a$ , on a horizontal plane through the crown. As a consequence, a correction to  $P_{30}$  would be applicable only if the arching ratio at a given depth varied significantly from the erching ratio at zero depth. This does not occur for the cylinders tested as Fig. 6.5 indicates (although this indication is not conclusive because of the scatter in data for these shallow burials). Hence  $p_a$  and  $P_{so}$  were considered interchangeable.

This does not negate the facts that the arching ratios do differ from group to group at zero depth of burial, and that the arching ratio at zero depth is not necessarily 1.0. For any study of the arching ratio for real structures at depth, it would be necessary first to study the response of the structure at zero depth where a known loading exists. Apparently,

load can be dissipated between the level of the crown and the level of the spring line.

6.4 Ultimate Strength

The collapse pressure, P sof , is plotted in Fig. 6.6 with respect to the stiffness parameter  $\frac{EI}{R^3}$ . The tests of the present investigation, Table 5.1, cover only a small part of the practical range of stiffness and pressure. In order to make the picture as complete as possible, results of other investigations in dense, dry sand are also indicated. The depth of burial is listed next to the symbol in terms of the cylinder diameter, d .

A dotted line indicates the yield value of a high-strength steel in hoop compression for a smooth cylinder. This establishes the upper bound limit of applicability of the elastic buckling theory and hence defines the area of concern for elastic buckling. Above this line the membrane response is inelastic and would be treated in terms of a ductility factor rather than stiffness.

In Figs. 6.7 through 6.10, the collapse pressure has been formed into a nondimensional parameter,  $P_{sof}R^3/EI$ . The test results are plotted in this form with respect to  $\frac{EI}{D^3}$ . A different set of theoretical equations is shown in each of Figs. 6.7 through 6.10. It was mentioned in Chapter 3 that the theoretical equations all contain the cylinder stiffness parameter,  $\frac{EI}{23}$ , as an independent variable.

Open symbols in Figs. 6.6 through 6.10 refer to tests which did not result in failure. Although these tests do not indicate the pressure at which the cylinder would have failed, they are pertinent because they do document areas where failure did not occur.

The amount of data available with which to correlate the clay

results is very slight. Luscher and Hoeg (1964, p 231) reported a series,  $\frac{\text{EI}}{\text{R}^3} = 0.011$ , that experienced failure very similar to their sand tests which were two orders of magnitude higher than the theoretical pressure predicted by the hydrostatic equation,  $p_0 = 3 \frac{\text{EI}}{\text{R}^3}$ , Fig. 6.7. The results of the present investigation,  $\frac{\text{EI}}{\text{R}^3} = 45$ , indicate that the failure pressure for cylinders in clay increases very slowly with increasing depth of burial, Fig. 5.48. The hydrostatic equation is in reasonable agreement with these results, Fig. 6.7, and the results of a test on a stiffer cylinder,  $\frac{\text{EI}}{\text{R}^3} = 82$ , conducted by Dorris and Albritton (1965). On the basis of this, it appears that the hydrostatic buckling equation should be retained for claylike soil media until such time as more experimental evidence fills in the gap between the available data points.

Although far from complete, the data available from tests in dense, dry sand are more plentiful. The present investigation in dense sand showed considerable increase in failure pressure with increase in depth of burial down to d/8, Fig. 5.48. Below this depth failure could not be precipitated with the pressure available, 500 psi. Donnellan (1964, p 42) experienced failures a. d/8 but none at d/4 at 160 psi. However, the conclusion that below some critical depth in dense sand, elastic buckling will not occur is precluded by the results of Bulson (1962) and Luscher and Höeg (1964). But, this conclusion may very well apply to cylinders which are stiffer than some critical stiffness.

The theoretical analysis developed by Luscher and Hoeg (1964, p 143), equation 3.16, is plotted in Fig. 6.8 for several depths of burial. It takes into account the change in soil stiffness with depth and pressure, and predicts the possibility of elastic buckling at depths greater than

d/4 for very flexible structures. The equation fits the author's experimental data and that reported by Bulson (1962) fairly well. For depths greater than d/4, the equation indicates that buckling will not occur before yield of the material for the cylinders used in the present investigation. Hence, this appears to be potentially an adequate design equation for interpolation.

However, for extrapolation of the data a much more conservative approach is in order. A lower bound for these data at zero depth of burial is established by equation 3.8, Fig. 6.9. Substituting  $k_z R = 400$ in equation 3.8,

$$p_c = 40\sqrt{\frac{EI}{R^3}}$$
, psi 6.6

where E is in units of psi, I is in units of in.<sup>3</sup> and R is in units of in. Although the theoretical equation has the hydrostatic buckling value,  $3\frac{EI}{R^3}$ , as a lower bound, it is not possible to say that this would be true for the actual conditions. For a stiff cylinder at very shallow burial, the soil could be a less desirable environment than water because of the nomuniform loading occurring through the soil.

Equation 3.8 with  $k_{z}^{R} = 1400$  fits the writer's data at d/8, Fig. 6.9, and is a lower bound to the data available for more flexible cylinders. Hence, it appears that

$$P_c = 75 \sqrt{\frac{BT}{R^3}}$$
, psi

6.7

would provide a more realistic lower bound to the buckling value than the hydrostatic equation used alone. The units are the same as in equation 6.6.

It is evident that the foregoing values of  $k_z$  are much smaller than those calculated for sand from the lows Formula in Section 6.2.

Equation 3.8 with  $k_{z}R = 37,000$  fits the d/8 no-failure data of the flexible cylinders, and is also shown in Fig. 6.9. It is possible that this may be an appropriate equation for the high overpressure region. This value of  $k_{z}R$  is still lower than those calculated from equation 6.2. If  $k_{z}R =$ 37,000 or higher, it is apparent that buckling will not occur before yield for many practical values of cylinder stiffness (greater than about 1.7) when the cylinder is buried at a depth below d/8. Hence, the theoretical variation of the dense sand properties with respect to pressure may be important only for design pressures below about 500 psi.

The theoretical equation, 3.17, which utilizes Poisson's ratio and Young's modulus of the soil is plotted in Fig. 6.10 for comparison. It follows the general trend of the available test data, but no definite conclusions can be drawn.

A comparison of the results of the A and B groups, Table 5.1, indicates that the cylinder strength may play a part in the buckling values. This is probably a reflection of the decrease in effective buckling stiffness which occurs when part of the cross section yields. However, the failure values between the groups did not differ by more than 25 percent although the yield values varied by a factor of 2.

The catastrophic manner in which the cylinders failed is probably a consequence of the large amount of strain energy in the cylinders at incipient collapse. Figs. 5.44 and 5.45 depict the failed cylinders. The irregularities in the postbuckling shapes were caused by the cylinder crowns striking the longitudinal rods (which connected the end caps) as they caved in. The postcollapse configuration in city is shown in Fig. 5.47.

CHAPTER 7. SUMMARY, CONCLUSIONS, AND RECOMMENDATIONS

7.1 Summery

Forty-six, small, horizontally oriented cylinders were tested in two kinds of soil media: dense, dry sand and stiff clay. The applied overpressure, vertical, and horizontal diameter changes for the static tests and hoop strains were measured. The cylinders were all made of aluminum. Three alloys were involved having yield stress values of 7,500, 12,700, and 42,100 psi. The cylinders had identical outside diameters of 3.5 in. and two thicknesses, 0.022 and 0.065 in. Hence, the cylinder stiffnesses,  $\frac{\text{EI}}{r_3}$ , were 1.7 and 45 (d/h = 159 and 54), respectively.

The test structures were buried at depths ranging from zero to three-quarters of the outside diameter, 2-5/8 in. Three overpressure rise times were used: a static rise time (10 to 15 min), a rapid rise time (13 msec), and a dynamic rise time (0.3 msec).

The relations between stress, thrust, moment, and diameter change were plotted and analyzed with respect to the surface overpressure. The pressure necessary to cause collapse was established and compared with several theoretical solutions and with the results of other investigations. The horisontal and vertical stiffnesses as indicated by the diameter changes were analyzed and compared with theoretical concepts.

It was not possible to collapse cylinders of either stiffness when buried in sand at depths equal to or greater than one-eighth the diameter, 7/16 in., under the available 500-psi pressure. In stiff clay, however, it was possible to define collapse even at the deepest burial, three-quarters of the diameter or 2-5/8 in.

## 7.2 Conclusions

All of the conclusions are based on the assumption of the planewave loading which was used during this investigation.

# 7.2.1 Cylinders in Dense, Dry Sand

The difference between static and rapid loading in the elastic response of the cylinder is small (within 20 percent). The rapid loading was observed, Figs. 6.1 through 6.3, to cause larger thrusts.

Inelastic strains are much higher under dynamic loading than under static or rapid loading at the same pressure, Fig. 5.36. However, a cylinder buried at a depth greater than one-eighth its diameter can sustain large inelastic bending strains without experiencing structural failure or collapse.

Based on an equivalent elastoplastic resistance function for the cylinder and an approximate step-pulse loading, a ductility factor of about 7 was found to be conservative for the dynamic tests. No failures occurred, so it is not possible to say what the ductility factor to define failure would be.

Thrust is generally a linear function of surface overpressure. It is largest at the spring line, smaller at the crown, and smallest at the invert. For overpressures greater than 200 psi, the average value of the horizontal force divided by the vertical force on the cylinder is about 0.8. However, the hoop compression theory appears to be adequate for design.

Noment is generally a nonlinear function of surface overpressure. It tends to increase at a decreasing rate (probably governed by local arching from point to point around the circumference of the cylinder), until the cylinder material begins to yield. Thereafter, the moments tend to decrease. The moments are larger in the stiffer cylinders. A depth of burial of one-eighth the diameter is a critical depth for the sign of the crown moment. At shallower depths the curvature increases, whereas for deeper depths the moment is positive and the curvature tends to decrease.

For zero depth of burial, the pressure to cause buckling failure can be defined by

$$p_{cr} = 40 \sqrt{\frac{EI}{R^3}}$$
, psi 7.1

where E is in units of psi, I is in units of in.<sup>3</sup>, and R is in units of in. This is an empirical fit of equation 3.8 to the test data with  $k_z R = 400$ , Fig. 6.9. For depths of burial equal to or greater than oneeighth the diameter, the pressure to cause buckling failure can be bounded until more experimental data becomes available by

$$\mathbf{p}_{cr} = 75 \sqrt{\frac{BI}{R^3}}$$
 7.2

where the units are the same as those in equation 7.1. This is equation 3.8 with  $k_R = 1400$ . Failure occurs (at the shallow burial) by a sudden snap-through of the crown. The result is a complete collapse. But, no collapse could be induced at depths greater than one-eighth the diameter for  $\frac{EI}{-3} \ge 1.7$  for pressures up to 500 psi.

Depths of burial greater than one-eighth the dismeter probably have more significant effects (on elastic buckling) than indicated by the allowable pressures from equation 7.2. However, since the effects of the depths were not satisfactorily defined because no failures occurred, they can only be considered as an additional factor of safety. Equation 7.2 represents points where no failure occurred and does not define failure.

However, this is a more realistic equation than the hydrostatic prediction. It is hypothesized that equation 7.2 is still overly conservative for values of  $\frac{EI}{D3}$  greater than about 1.7.

It is not possible at present to identify adequately the appropriate soil properties controlling cylinder collapse with soil properties obtained from standard laboratory tests.

The technique used to place the sand in the vicinity of the cylinder can affect the response of the cylinder and apparent deformation stiffness by as much as 50 percent. However, the pressure required to cause collapse differs by only  $\pm 25$  percent. Sprinkling in the vicinity of the cylinder is less effective than vibrating or rodding.

The arching ratio (defined as the average spring-line thrust divided by the overpressure times the radius) for cylinders buried with the crown tangent to the soil surface is not necessarily 1.0.

7.2.2 Cylinders in Stiff Clay

Collapse of the cylinder occurs by a sudden snap-through of the crown. Regardless of the depth of burial, this mode of failure occurs even at the maximum depth tested, three-quarters of the diameter.

Only a small increase in failure pressure results from an increase in depth of burial. The hydrostatic buckling equation

$$P_{cr} = 3 \frac{KI}{R^3}$$

was appropriate for the cylinders used, Fig. 6.7, and should be slightly conservative for cylinders buried at depths greater than one-eighth the diameter. This equation implies a low value of  $k_{\rm s}R$ .

Noments and deformations of the cylinder were much larger than

67

7.3

in sand at comparable pressures. They were both highly nonlinear functions of pressure.

7.3 Recommendations for Future Study

High pressure tests (500 psi or greater) should be conducted in dense sand with cylinder stiffnesses,  $\frac{EI}{R^3}$ , between 0.1 and 1.0 for the purpose of establishing failure pressures for depths of burial greater than one-eighth the cylinder diameter. Materials with high yield strengths, such as high-strength steel or aluminum, would best serve the purpose. Elastic buckling could be isolated relative to the buckling stiffness without consideration of the reduced stiffness due to yielding.

Some ultimate strength tests should be conducted with relatively large (2-ft-diameter) cylinders in the WES Large Blast Load Generator to investigate the possibility of size effects. These should have the same value of  $\frac{\text{EI}}{\text{R}^3}$  as some smaller diameter cylinders discussed in the literature, or else small companion cylinders should be tested concurrently.

A cylinder with  $\frac{51}{R^3}$  = 220 should be tested at zero depth of burial in dense sand at pressure greater than 500 psi to extend the range of knowledge of equation 7.1.

The work on elastic buckling should be done with static loading (but fast enough that longtime effects such as creep do not enter) to gain the most for the least cost. Selective dynamic testing should then be done to assure the applicability of the knowledge gained from the static tests.

Once the limits of the buckling problem are established, then dynamic studies should be conducted to determine an appropriate magnitude for the ductility factor to define collapse in the nonelastic region of

cylinder response. Since yielding is not a proper criterion for failure, it is doubtful that the studies of the elastic response of cylinders will shed much light on the ultimate strength except when buckling governs.

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Once the dense, dry sand-cylinder interaction is fully understood, other soil environments such as medium density (relative density of 50 percent), and partially saturated sands should be investigated. It may then be possible to develop a single equation which can take into account the significant soil properties in a realistic manner.

Concurrent with the foregoing, an attempt should be made to determine the pressure distribution on the surface of the buried cylinder from the measured strains. The solution by Riley (1965) for WES which expresses the load in a Fourier series with undetermined coefficients could be used.

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Table 4.1

Geometric and Material Properties of Test Cylinders  $\mathbf{E} = 10 \times 10^6$  psi; t = 10.5 in.; d = 3.5 in.; t/R = 6

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159	1.7	8.9	8	1.74	0.022	45,100	37,500	94-1909đ	U
54	45.0	228.9	2289	1.72	0.065	12,700	л,œ	D5052-0	
5	45.0	228.9	5289	1.72	0.065	0012	4500	0-1909d	<b>M</b>
ħ	45.0	228.9	5589	1.72	0.065	6100	3400	0-1909d	A
ħ	45.0	228.9	5389	21.72	0.065	892	8	D-19091	<b>.</b>
নাম	Bail)	( :या-वा) म	I (10 <sup>-8</sup> 1n. <sup>3</sup> )	R (1a.)	h.)	aye (Jac)	(j.1)	Alumian	Croug
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	•••		Z* * 0 in.	0 <u>in</u> .	2 * 2/16 in	16 fn.	7 = 5/	76 45	16 = 6	76 45 7	14 - 4	4.5		1-2/1 4-	0	
(Jrown)			3	2 - 2 -	5	0 9	12	<b>6</b>			-175	e co	-120	P So	CA] -	P S S
							Janer	<b>D</b> 81	Inder	1sd	Inder	<b>P81</b>	inder	1sd	inder	Del
	•	011110	Į	22	~	í A			<b>A-</b> 2	##005	A-3	**00ú	A-4	5:00		
	Send	Rapid	10	8	6-v	8			<b>A-</b> 8	\$005	A-7	\$00 <b>*</b> *	<b>A-6</b>	2004		
~	Send 5	Static	ц.	315					B-5	¥#005	<b>B-</b> 2	500##	B-3	500##	B-4	2004
44	Sand I	Rapid	9	8					B-7	¥*005	8-8	500##	<b>Р</b> 9	<b>**00</b> 5	<b>B-1</b> 0	2004
***	Send .	Static	1-5	8	7-5	195	C-5	500##	C-2	500##	0-3	500 <b>*</b> *				
-	Sand	Reptd	G-6	13	2-3	350	G-8	200	6-2	500	C-10	200##				
~	Sand S	Static	E-3	33					<b>E-</b> 2	140044	E-1	140++				
	Send	Dynamic	8-6	1150					E-5	262**	E-4	264**				
U I		Static	ц Ч	8	•				D-2	130	D-3	100	D-4	190	D-5	180
Q.	Clev D	Dynamic	•								D-10	**26	<b>D-</b> 6	160**		
			•								<b>D-</b> 8	116**	D-7	180		
											<b>D-9</b>	148				
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			Table Sec					
hrust,	Noment .	rad	Deflection;	jests	A-1.	A-2, A-3A	. A-38. A	<del>~</del> 5
			- On	erpres	sure,			
	150	2	<u> </u>		<u> </u>	<u> </u>	<u></u>	

<u></u>						Overp	resqure,	
Cage	Neenur villent.	50	100	150	20	250	-1-	<u></u>
							1	
						12 - 0 10		
1	Strain, win./in.	194	396	485	-85	276	-543	
	Stress, psi	19 <b>4</b> 0	3960	4850	4850	2760 1862	-5102	
1.	Strain, µin./in.	20 200	53 500	145 1850	400 4080	7147		
1-14	Stress, psi Thrust, 1b/in.	70	146	205	290	3		
1-14	Namet, in1b/in.	-0.61	-1.21	+1.20	-0.27	1	••	
	Strain, win./in.	44	88	141	141	132	123	
3	Stress, pel	440	860	1410	1610	1320	1230	
34	Strain, win./in.	100		240	327	بلوية	NÔC -	
50	Stress, pai		17-0	2400	3270	4540	4800	
3-34	Thrust, 10/in.	47	85	126	152	190	196	
	Moment, inib/in.	0.20	0.30	0.35	0. <b>65</b>	1.13		
2	Strain, sin./in.	1 <b>8</b> 3	319	475	500	1440	1040	
	Stress, pel	_ <b>83</b> 0	3190	1750	5864 106	<b>68</b> 07 550	7125	
26	Strain, uis./is.	-18 -180	24 140	560	1080	222	6022	
2-2	Stress, psi Thrust, 10/18.	54	106	173	268	NO2	434	
	Humant, in1b/in.	-0.71	-1.0/	-1.48	-1.79	~0.53	-0.31	
4	Strain, wim /im.	125	224	344	581	10+1	1231	
•	Stress, pai	1250	2240	مشتو	5350	6200	6623	
44	Strain, win./in.	9	51	102	191	404	510	
	Stress, pel	90	510	1020	1910	NONO 263	5004 308	
la-lag	Thrust, 1b/in.	44 - 0 A1	-0.61	145 -0.85	248 +1.29	363 -0.54	յաս +0.էԳ	
	Nomat, in1b/in.	-0.41					-	
DC1	Deflection, is.	0.001	0.001	0.003 0.002	0.007 0.003	0.017 3.005	0.037 0.007	
DC2	Deflection, in.	0.001 59	0.000	165	221	291	••	
1-1413-36 2-2616-66	Avg thrust, lb/in. Avg thrust, lb/in.	19	99	159	258	363	411	
a - 436 ( 4+ <b>46</b>	a and conservation that the	1.20	1.17	1.04	9.86	0.76	••	
	•				<b>-</b>	- 1	(	
						5 (2 + 3/10		
1	Strain, sin.,	290	953	632	35 <b>4</b>	23		-515
-	Strear, pti	2900	5236	5568	5240	2360		-5062 6820
la	Strain, Mais./in.	0	0	133	101 1010	1962 7201		9505
	Stress, pai	0 34	0 179	<b>173</b> 0	307	391		445
1-16	Thrust lb/in. Noment, inlb/iz.	1.08	-1.98	-1.59	-0.55	1.27		3.03
		161		281	256	275		262
3	Streig, pio./in.	1010	23é 2360	2010	2980	2720		2620
34	Stress, pti Streis, pix/is-		10		106	182		259
<b>3</b> 74 0	Street, pti	100	100	96 960	1060	1820		2590
3+34	Thrust, 1b/is.		80	10	- UT	749		-0.91
	Numme, in16/in.	-Q. 🏍	-0.80	-0.85	0.68	-0. 32		
2	Strain, win /in.	113	241	356	532	1005		2145
	Manas, pel	1130	2430	3560	1436	6285		7301
<b>i</b> n	Strain, sin./in-	0	. 31	1	16h0	523 5108		634.2
• •	Street, pti	0 37	310	· 780	200	375		ingle.
3-23	Thrust, 15/is. Memori, 1815/16.	-0.6	.0.75	-1.00	-1.29	-0.35		•Ð.13
		162	34	518	111	12%0		4217
•	Strein, sin /in- Strein, phi	1620	1	3000	1824	6624		7341
-	Strein, pin /in	••••	-11	ð.	54	329		1162
	#1 POSE, 111	-130	-110	Ŷ	\$30	3490		6501
3-44 ···	Durant, Myin.	- <b>-</b>	108	34	<b>N</b>	366		194. 18.10
	Hundl, LASSIN/IR.	~ (1)	÷.₽	-1. <b>8</b> 8	-2.00	•••- <b>*</b>		
<b>9</b>	Deflection, in.	-2.301	-0.007	0.001	0.005	0.017		ହି: ୁ <b>ରୁ</b> ଅ.ସେମ୍ପ
202	beriantian, in-	0.003	¢.001		0.00	12.000 10.000		307
- iai )- )a	Any thrust, 16/18	R	L)E	17)		340		459
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							1.1	
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		194	397	307	596	550		
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¥4	BEPRIN, MIE /IN	2		192	338	913 1004		101
	Stress, Sti		100	15304 194	110	1		¥1
in in	Direct, 35/18.	. 0 🐝	+1.17	-4.6	4.7	÷.89		* 11
. <u>.</u>		ی در در در در از		<b>3</b> 0	Q.			M
3	Pireis, pis./10:							3003
	Maria, ala fia		NT NT		1			33 <b>4</b> .
	Figure, sei	10	200	900	Midt	-	1999. 1999	1140
3-34	Parent, Hyla.	57	107	14.3		<b>20</b>		
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-	Staves, ret		- 200		2200			\$ <b>#</b> # <b>J</b> *
Surlag.	Purvet, Mylas	65	117	-1.53		- <b> </b>		**
	Summer, in	-4 🦋	+1.42					
10	buffaction, in	¢	6.000	9. <b>101</b>	9.434Ú	9.089		5 <b>012</b> 9.945
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					⊀abte (	(Concluded)					
Gege	Measurement	50	100	150	200	Overpressu 250 27		325 350	400	450	500
						<u>3A (Z = 7/8 1a.)</u>					
1	Strein, µin./in. Stress, poi	<del>88</del> 380	169 1690	5130 513		213 2130	389 3890	940 5934	1362 6745		
18.	Strain, µin./in. Stress, psi	87 870	174 1740	289 2890	<b>463</b> 4630	820 5899	1 <b>38</b> 0 6759	2?61 7365	3125 7033		
1-18	Thrust, 15/in. Moment, in16/in.	57 0.00	113 0.92	163 0.27	215 0.93	300 1 <b>.36</b>	. 353 0.50	444 0.47	477 0.37		
3	Strain, ui. /in. Stress, psi	166 1660	250 2800	367 3670	464 4640	560 5267	647 5595	796 5857	954 6135		
3 <b>a</b>	Strain, µin. An. Stress, _si	31 310	55 550	80 800	128 1290	185 1850	<b>355</b> 0	332 3320	473 4 <b>73</b> 0		
3-34	Thrust, 15/in. Moment, in -1b/in.	64 -0.48	-0.79	145 -1.01	192	240	273 -1.26	327 -0.86	367 -0.41		
2	Strain, µin./in.	171	308	426	564	1450	2680	4150			
24	Stress, psi Strein, µin./in.	1719 -14	3080 26	4260 110	5285 262	6815 1170	7593 2341	8337 3745			
2-2n	Stress, psi Thrust, 1b/in.	-140 51	26) 109	1100	2620 266	6515 435	7409 489	8138 505			
4	Noment, in1b/in. Strain, µin./in.	-0.65 213	-0.99 368	-1.11 530	-0 697	-00 1198	-0.06 2243	-0.07 3545	 4600		
4.8	Stress, psi Strein, µin./in.	2130 •46	3680 -20	5133 33	5683 113	6564 435	7355 143 <sup>1</sup>	8040 2601	8558 3601		
4-40	Stress, psi Thrust, lb/in.	-460 54	-200 133	330 183	1130 250	4350	6802 462	7550	8067 540		
	Noment, in1b/in.	-0.91	-1.37	-1.74	-1.73	-0.62	-0.20	-0.17	-0.17		
DC1 DC2	Deflection, in. Deflection, in.	0.005 0.004	0.007 0.006	0.009	0.013	C.021 0.011	0.029 0:013	0.039	0.048		
1-12:3-32 2-22:4-42	Avg thrust, 1b/in. Avg thrust, 1b/in.	61 53	110 111	154. 179	204 258	270 407	320 475	386 521	422 540		
	đ	1.15	0:99	0,86	0.79	0.66 3B_(L = 7/8 in.)	0.69	0.74	0.75		
1	Strain, µin./in.	-206	-244	-264	-257	-264	-198	_ أيفنا	6é	294	896
la	Stress, psi Strain, µin./in.	-2060 338	-2440 521	-2640 695	-2570 840	-2640 1070	-1980 1331	-440 1689	660 2048	2940 2620	6 <b>03</b> 3 3702
1 <b>-1a</b>	Stress, psi Thrust, lb/in	3380 43	5093 90	5679 132	5934 167	6339 205	6720 258	7005 327	7248 360	7561 422	8117 476
3	Moment, in1b/in. Strain, µin./in.	1.92 79	2.69 122	3.16 175	3.27 21c	3.35 271	3.05 306	2.32 376	1.52 50 <b>7</b>	1.11 655	0.64 814
3a.	Stress, psi Strain, µin./in.	790 55	1220 117	1750 184	2180 234	2710 296	3060 362	3760	5031 621	5609 817	5889 1056
3-3a	Stress, ps1 Thrust, lb/in.	550 44	1170 78	1840 117	2340 14.7	2960 184	3620 217	4740 276	5530 344	5894 374	6315 397
	Moment, in10/in.	-0.08	-0.02	0.03	0.06	U.09	0.20	0.35	0.18	0.19	0.15
2	Strain, µir./in Stress, Isi	216 2160	368 3680	518 5080	650 5600	800 5864	920 6075	1200 6568	2570 <b>7</b> 534	4010 8268	
28	Strain, µin./in. Stress, psi	<b>69</b> 69ට	131 1310	220 2000	302 3020	392 3920	488 4880	742 5762	1971 7206	3302 7920	4420 8470
2-24	Thrust, b/in. Moment, in10/in.	93 -0.52	162 -0.83	240 -1.C4	298 -0.95	341 -0,62	<b>366</b> -0.37	401 -0.28	479 -0.12	526 -0.12	
4	Strain, µin./in. Streas, psi	199 1990	33ð 3380	468 4680	590 5400	736 5751	860 59 <b>7</b> 0	1020 6251	1898 7166	2842 7680	4110 8 <b>31</b> 7
44	Strain, µin./in. Stress, psi	47 470	100	174 1740	234 2340	315 3150	368 3680	515 5067	1375 6755	2300 7386	3362 7950
<b>կ</b> կ <b>a</b>	Thrust, 1b/in. Moment, in1b/in.	80 -0.54	142 -0.84	209 - 1.04	264 -)14	315 -0.93	346 -0.65	375 -0.37	453 -0.15	490 -0.10	529 -0.13
DC1.	Deflection, in.	0.007	3.011	0.013	0.016	0,020	0.022	0.026	0.034	0.043	0.053
DC2 1-1a:3-3a	Peflectica, in. Avg thrust, 1b/in.	0.004 44 87	0.007 84	0.008	0.009	0.011	0.013	0.014	0.015 357 466	0.016 398	0.017 437
2-2a:4-4a	Avg thrust, 1b/in. 9	0.51	152 0,55	225 0.56	281 0.56	<b>326</b> 0.59	355 0.67	388 0.78	466 0.7?	508 0.78	529 0.83
	Church into lin		. 20			(Z = 1-3/4 in.)			<b>-</b>		
1	Strain, µin./in. Stress, psi	0 0 134	39 390 201	78 780 205	143 1430 443	195 1950	295 2950	507 5031	780 5829	1142 6466	1535 6883
la 1-le	Strain, µin./in. Stress, psi Thrust 1b/in	1340 1340 44	2010	295 2950	4430	531 1567	900 6040	1290 6688	1706 7019	2226 7346	2760 7636
1-1 <b>6</b>	Thrust, 1b/in. Moment, in1b/in.	0.47	78 0.57	)21 0.76	1,06	261 1.35	330	391 0.53	425	454 0.29	474 0.26
3	Strain, µin./in. Stress, pai	166 1660	280 2800	402 4020	543 5191	744 5 <b>76</b> 5	9 <b>63</b> 6151	1226 6614	1480 6839	1813 7104	2035 7241
34	Strain, µin./an. Strass, psi	-38 -380	-57 -570	-76 -760	-114 -1140	-123 -1230	-123 -1230	-104 -1040	-86 -860	-3d -380	38 380
3-38	Thrust, ib/in. Moment, inib/in.	42 -0,72	72 -1.19	106 -1.68	139 -2.30	186 -2.68	220 -2.76	269 -2.67	301	337	- 364 -1.93
5	Strain, µin./in.	267 2670	445 4450	609 5485	807	1275	2140	3322	4470	5 <b>65</b> 0	6900
28	Stress, psi Strain, µin./in.	-41 -41	-41	-10	5876 10	6676 134	7298 909	7930 2029	8494 3043	9023 1110	95.58 52140
2-24	Stress, psi Thrust, 1b/in. Moment in alb/in.	-1.08	-410 131 -1:71	-100 191 -2.08	100 241 -2 10	1340 331	6056 443	7236 493	7789 529	8317 565	88.ji 98.
4	Moment, in1b/in. Strain, µin./in.	194	-1,71 296	445	-2.19 571	-1.68 765	-0.41 1267	-0.25 22 <b>38</b>	-0.25 3220	-0.25 4293	-0, 24 54, 24
44	Stress, psi Strein, µin./in.	1940 -22	2960 -11	4450 34	5316 122	5802 290	6670 793	7352	7880 2688	84C7 3707	895 n 472 5
4-42	Stress, psi Thrust, 1b/in.	-220 56	-110	540 156	1220 223	2900 313	5852 407	7083 470	7597 503	8119 537	862G
DCA	Noment, in1b/in. Deflection, in.	-0.76 0.002	93 -1.08 0.003	-1.45	-1.52	-1.03	-0.29	-0.09	-0.10	-0.10	-0.11
DC2 1-1a:3-3a	Neflection, in. Avg thrust, 1b/in.	0.004	0.003 0.005 75	0.005	0.007	0.010	0.016 0.010	0.021 0.011	0.028	0.034	0.044
2-2414-44	Avg thrust, 1b/in.	65 0.66	115	114 174	165 232	225 322	279 425	330 482	363	396 551	419 585
	1	0.00	0.67	0 <b>.66</b>	0.71	0.70	0.66	0.68	0.70	0.72	0.72

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Table 5.3 Strain, Stress, Thrust, and Moment; Tests A-6, A-7, A-6, A-9, A-10

											<u></u>
Gage_	Messurement		100	150	0v _200	erpressure,	<u>100</u>	350	400	450	500
				Test	A-10 (Z = 0	in.)	~				
	Strain, µin./in.	-149 -14 Ю	432 4320	894 6029	12 <b>8</b> 1 6681	461 4610	-1830 -8676				-
	Stress, psi Strein, µin./in.	301	51	26	309	3942 8235	10400 10950	-			
14	Stress, psi Thrust, lb/in.	3090 52	510 157	260 259	3090 365	465	يلوخ		-		
	Moment, inb/in.	1.61	+1.34	-2.14	+1.02 466	0.95 423	8.24 503				
	Strain, µin./in. Stress, pui	-80 -800	151 1910	330 3330	4460	4230	5013				
	Strain, µin./in. Stress, poi	231 2310	284: 2840	32 <b>6</b> 32 <b>6</b> 0	389 3890	45 <b>8</b> 45 <b>8</b> 0	503 5013		-		
38.	Thrust, ib/in. Noment, inib/in.	1.09 1.09	141 0.47	214 -0.01	271 -0.20	2 <b>86</b> 0 <b>.</b> 12	32€ ∂₊00				
	Strain, µin./in.	269	316	398	457	1630	2662				
	Stress, poi Strain, µin./iu.	2690 +12	3160 194	3980 198	4570 349	6958 1235	7691 1947				
24	Stress, psi Thrust, 1b/in.	-120 84	1040 137	1580 194	3450	6630 442	7193 484				
FOA N	Moment, in1b/in.	-0.99	-0.75	-0.70	-0.38	-0.11	-0.18				
	Strain, µin./in.	275 2750	375 3750	462 4620	713 5711	13 <b>38</b> 6726	2564 7531				
	Stress, psi Strain, µin./in.	-11	80	159	353	1014 6241	1276 6677				
4.18	Stress, psi Thrust, lb/in.	-110 86	800 148	1590 202	3530 322	423	464				
	Moment, in1b/in.	-1.01 51	-1.04 149	-1.07	-0.77 318	-0.18 376	-0.30				
3 <b>a:3-3a</b> 3a.4-43	Avg thrust, lb/in. Avg thrust, lb/in.	ê5	143	237 195	292	433	474				
	4	0.60	1.04	1.20	1.09	0.87					
	Strain, µin./in.	-207	445	<u>Test /</u> 1482	<u>-9 (7 = 3/16</u> 2667	<u>2 19. )</u> 2445	1748	1185	-1778		
	Stress, psi	-2070	4450	6840	7586 81	7466 1462	7052 5982	6542 10042	-7076		
	Strain, µir./in. Stress, psi	<b>3</b> 24 3240	162 1620	- 162 - 1620	810	ú <b>8</b> 25	9150	10830	••		• *
la	Thruat, 1b/in. Moment, in1b/in.	38 1.87	197 -1.00	283 -2.86	398 -1.68	466 -0.22	531 0.74	580 1.42			
	Ctrain, µin./in.	35	263	518	658	702	676	711	737		
	Stress, psi Strain, µin./in.	350 131	2630 98	508୦ 8ନ	5614 98	5692 155	5646	5707 262	5753 311		<u>.</u>
31	Stress, psi Thrust, 1b/in.	1310	9 <b>8</b> 0 117	. 820	9 <b>8</b> 0 237	1550 264	2210 279	2620	3110 314		
	Moment, in1b/in.	0.34	-0.58	-1.53	-1.75	-1.57	-1.29	-1.14	-0.94		
	Strain, µin./in. Stress, psi	2410 2410	289 2390	361 3610	602 5454	1133 6450	2699 7603	3687 8109	4892 8702		
	Strain, µin./in.	25 250	0	199 1996	373 3730	972 9819	2414 7449	3634 8083	4505		~
2 <b>a</b> ,	Stress, psi Thr st, 1b/in.	86	9 <b>4</b>	182	309	399 -(-,22	489	526 -0.01	559 -0.07	÷	• •
	Moment, in1b/in. Strain, µin./in.	-0. <b>7</b> 6 330	-1.02 356	-0.57 559	839	1444	3433	4399	5670		
	Stress, psi	330	3560	5262 74	5933 221	6814 959	79 <b>8</b> 4 2754	8559 3511	9031		
	Strain, µin./in. Stress, poi	-490	0	740	2210	6144	7633	8170	8042		
18	Thrust, 10/in. Moment, in1b/in.	91 -1.33	116 -1.25	204 -1.67	304 -1,32	425 -C.24	508 -0.12	540 -0.10	575 -0.13		
la:3-3a	Avg thrust, 15/in.	46	157	239	318	365	405	439	567		
2 <b>1:</b> 4-48	Ave thrust, 1b/in. q	89 0.52	105 1.50	1:3 1.24	307 ⊥.04	412 0.89	- 499 9 <b>.81</b>	533 0. <b>8</b> 2			÷.,
				Test.	A-B (2 = 7/1	6 in.)					<u>.</u>
	Strain, µin./in.	-220	422	906 6053	1293	1188	1139 6454	1355 6739	1557 6900	1698 7012	1707 7019
	Stress, psi Strain, µin./in.	-2200 249	4220 -28	6051 166	6690 443	6547 1773	3574	5154	7094	9726	12303
14	Stress, psi Thrust, 1b/in.	2490 9	-280 128	1660 299	4430 38=	7072 444	8054 430	8818 511	9618 545	10703 584	11569
	Moment, in1b/in.	1.65	-1.58	-1.55	-0.64	0.17	0.52	0.71	0.93 2627	1.26	1.50
	Strain, µin./in. Stress, psi	041 1460	474	849 5950	1236 6631	1529 6:78	1804 7097	2133 7295	2437 7461	~- 	••
<b>.</b> .	Strain, µin./in.	89 890	33 330	17 170	500 500	127 1270	283 2630	371 3710	533 5147	749 5:774	100 <sup>1</sup> 622)
.ła	Thrust, 1b/in.	. 76	165	250	806 -2.02	349 -1.68	390 -1.14	414 -0.93	435	••	
	Moment, inib/in. Strain, µin./in.	-0.20 3 <b>8</b> 1	-1.55 429	-2.17 620	-2.02 787	2290	4080	5487	7348	9329	1133
	Stress, psi	3810	4290	5534 176	3841 427	7381 1586	8303 3297	8956 4581	9723	10539 7853	1125 956
a	Strain, uin./in. Stress, psi	25 250	125	1760	4270	6923	7918	8549	بلاويرو	9931 665	1063
-2 <b>8</b>	Thruct, 1b/in. Mement, in1b/in.	-1.25	1 <b>8</b> 0 -1.07	253 -1,41	347 -0.49	466 -0.16	527 -0.14	569 -0.14	615 -0.19	-0.21	-0.2
	Strain, µin /in.	380	456	608	836	2155	3677	4945	6568 9401	8242 10091	9 <b>89</b> 1077
L	Stress, psi Strain, µin./in.	3800	4360 100	54 <b>8</b> 0 201	5927 437	<b>73</b> 07 1413	8104 2926	8728 4305	5685	7367	861
- .48	Stress, psi Enrust, 1b/in.	0 123	1000 1 <b>8</b> 1	201.0 258	4370 353	6786 459	7726 515	8413 557	9037 599	9731 644	1024) 68
/18	Moment, in1b/in.	-1.34	-1.25	-1.29	-0.47	-0.19	-0.13	-0.11	-0.13	-0.13	-0.1
-1a:3-3a -20:4-4a	Avg thrust, 1b/in. Avg thrust, 1b/in.	43 128	147 181	275 256	347 350	397 463	435 521	4 <b>63</b> 563	490	655	69
	well entired to the	0.34	0.81	1.07	0.99	0.86	0.83	0.82	0.81		

(Continued)

## Table 5.3 (Concluded)

	Concerning the second s		and the second		Overpressur		_				
Hegerit went.	50	100	150	200	25/)	300	150	400	450	500	
			Test	A-7 (2 = 7/	<u>'6 in.)</u>						
Strain, uin./in	-155	103	435	675	766	831	1078	1389	1857	2338	
Stress, pet		1030	4150	5644	5804	5919		6767		7407	
Strein, uir./in.	251	275	461	769	1271	1547	2268	2972	3839	4365	
Stress, pei		2750						7751		8443	
Thrust, It/in.										516 0.36	
Formate, 19 Lty III.	13	0.01	0.10	0.00	0.52	0.37		0.34	0.37	0.50	
Strain, µin./in.	-44	61	217	330	391	400		530	626	u <b>78</b>	
Stress, pei										5649 1106	
Strain, pin./in.										6403	
Thrust, 1b/in.	46		161	206	249	268	324			392	
Moment, in1b/in.	0.61	0.65	r.21 3	-0.03	-0.05	0.09	0.12	0.18	0.15	0.27	
Strain, uin /in		371	1.36	613	1551	2650	4088	5510	6016	8047	
				5521	6895		8307		4545	10011	
		70	245	420	1348	2346	3799	5078	6426	7494	
Stress, pai	350	700	2450	4200	6734	7413	8164	8787	9343	9783	
Inruct, 16/in.										643	
Noment, in16/10.	-0.79	-1.06	-0.67	-0.49	-0.06	-0.06	-0.05	-0.00	-0.07	-0.08	
Strain, win./in.	262	459	<sup>0</sup> 689	1025	2510	3765	5410	7107	8991	10289	
Stress, pai	2820	4590	5669	6260	7501	8148	8924	9624	10400	10910	
Strain, µin./in.	-16		212	\$75						8681	
Stress, pai									9793	10272 689	
Thrust, 15/1D.										-0.23	
Manual, 1010/10.	~1.05	-1.30	-1.33	-0,44	-0.09	-0.19	-0.1)	-0.2)	-0.21	-0.23	
Avg thrust, 1b/in.	39	111	223	289	328	345	389	414	437	454	
				352				594		666 0.68	
4	0.43	0.11	0.09	0.02	0.11	5.09	0.11	0.10	0.09	0.00	
			Test A-	-6 (Z = 1-3/	4 in.)						
Strain, µin./in.	-90	-26	167	370	619	683	815	1057	1354	1793	
Stress, pai			1670							7088	
Strein, µin./in.										2831	
Stress, pti					6320					7674 480	
Moment, in .lk/in.			1.12	0.62						0.20	
Strain, µin./in.							1456			2675	
Stress, pol Strain win /in										7698 -606	
Stress, nai		900						-5142		-5471	
Thrust, 1b/in.	-37	77	110	140	183	201	2	239	264	292	
Moment, in. lb/in.	0.34	-0.20	-1.14	-2.21	-3.17	-3.63	-3.96	-4.25	-4.25	-4.11	
Strain, uin./in.	262	525	708	OSP	1389	2529	3500	5162	6144	7240	
	2620	5111		6100				8822	9227	2683	
Strain, µin./in.	-31	-41	123	260	743	1754	2774	4255	5215	6664	
Stress, pai										9446	
Thrust, 1b/in.										622	
NOMENT, IN10/1R.		-1.99	-1.70	-1.10	-0.37	-0.15	-0.15	-0.15	-0.13	-0.08	
Strein, µin./in.	199	399	487	612	1161	2288	3421	5113	6307	7890	
Stress, psi		3990		5498		7380				9951	
				305						6899 9542	
Through, jb/in.										634	
Moment, in1b/in.	-0.70	-1.17	-1.16	-0.90	-0.08	-0.04	-0.07	-0.10	-0.12	-0.14	
Are threat 1h/4-	<u>۵</u> ۵	<b>0</b> 1)	161 -	357	286	300	317	<b>2</b> 41.	262	386	
Avg thrust, 10/1n.	70	155	233	308	413	475	512	562	590	500	
	Streas, pai Strain, win./in. Strass, pai Thruet, lb/in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strain, win./in. Strass, pai Thruet, lb/in. Avg thrust, lb/in. Avg thrust, lb/in. Strain, win./in. Strass, pai Strain, win./in. Strass, pai	Strain, µin./in.       -155         Strean, µin./in.       -1550         Strean, µin./in.       2510         Strean, µin./in.       2510         Strean, µin./in.       143         Strean, µin./in.       143         Strean, µin./in.       143         Strean, µin./in.       143         Strean, µin./in.       145         Strean, µin./in.       145         Strean, µin./in.       145         Strean, µin./in.       258         Strean, µin./in.       259         Strean, µin./in.       250         Strean, µin./in.       2510         Strean, µin./in.       2510         Strean, µin./in.       2510         Strean, µin./in.       250         Strean, µin./in.       252         Strean, µin./in.       262         Strean, µin./in.       262         Strean, µin./in.       31         G       -300         Strean, µin./in.       90         Strean, µin./in.       90	Strain, µin./in.       -155       103         Strain, µin./in.       2510       275         Strain, µin./in.       2510       275         Strain, µin./in.       13       275         Strain, µin./in.       143       0.61         Strain, µin./in.       185       245         Strain, µin./in.       258       371         Strain, µin./in.       258       371         Strain, µin./in.       258       370         Strain, µin./in.       252       163         Strain, µin./in.       252       163         Strain, µin./in.       262       459         Strain, µin./in.       262       459         Strain, µin./in.       264       171         Nument, inlb/in.       -16       66         Strain, µin./in.       282       459         Strain, µin./in.       105       -1.38         Arg thrust, lb/in.       39       111         Arg thrust, lb/in.       105       <	Strain, win./in.         -155         103         415           Strain, win./in.         -1550         1030         4150           Strain, win./in.         2510         2750         461           Diracs, Tayin.         31         125         285           Strain, win./in.         .446         61         217           Strain, win./in.         .446         61         217           Strain, win./in.         .466         610         2170           Strain, win./in.         .466         610         2170           Strain, win./in.         .466         99         161           Noment, inlb/in.         0.61         0.65        213           Strain, win./in.         2500         3710         436           Strain, win./in.         350         700         2450           Strain, win./in.         262         459         689           Strain, win./in.         282         459         689           Strain, win./in.	Test A-7 (2 - 1/2)           Birnan, pai. /in.         -155         103         415         675           Birnan, pai. /in.         251         275         461         769           Birnan, pai.         2510         275         461         769           Birnan, pai.         2510         275         461         5009           Birnan, Min./In.         143         0.61         0.16         0.06           Birnan, Min./In.         146         217         330           Birnan, Min./In.         146         217         330           Birnan, Min./In.         145         245         277         305           Birnan, Min./In.         145         245         277         305           Birnan, Min./In.         258         371         436         662           Birnan, Min./In.         258         371         436         662           Birnan, Min./In.         259         143         221         332           Birnan, Min./In.         262         159         669         1025           Birnan, Min./In.         262         159         669         1025           Birnan, Min./In.         266         171         2	Test A-7 (2 - 7/8 in.)           Strein, pin./in.         -155         103         415         675         766           Strein, pin./in.         -1550         1030         4150         5644         5804           Strein, pin./in.         251         2750         4610         5809         6673           Strein, pin./in.         1.45         0.61         0.16         0.06         0.31           Strein, pin./in.         .440         61         2170         3300         391           Strein, pin./in.         .446         69         161         206         2470           Strein, pin./in./in.         .259         371         436         662         151           Strein, pin./in./in.         .259         370         245         420         1348           Strein, pin./in./in.         .259         669         1025         271         275           Strein, pin./i	Test A-7 (2 - 7/8 is.)           Bitrein, µis./in.         -155         103         k15         677         765         831           Bitrein, µis./in.         -155         1030         k150         5644         5904         5919           Bitrein, µis./in.         -150         2750         4610         5609         6673         6692           Bitrein, µis./in.         -143         0.61         0.16         0.06         0.31         0.35           Bitrein, µis./in.         -440         610         217         3300         3910         4000           Bitrein, µis./in.         185         255         277         305         3760         425           Bitrein, µis./in.         1850         2450         2770         3050         3760         425           Bitrein, µis./in.         1850         700         2450         1348         2246           Bitrein, µis./in.         350         700         2450         4260         4571         7724           Bitrein, µis./in.         95         143         221         332         443         487           Bitrein, µis./in.         10.79         -1.66         -0.67         -0.49         -0.06 <td< td=""><td>Event Ar 1 (2 = 7/8 in.)           Breads, pits./is.         -155         103         kits         675         766         831         1078           Breads, pits./is.         -155         103         kits         675         766         831         1077         2268           Breads, pits./is.         -1250         275         k610         5609         6673         6672         77969           Breads, pits./is.         1.43         0.61         0.16         0.06         0.33         0.35         0.33           Breads, pits./is.         1.43         0.61         0.16         0.06         0.31         0.35         0.33           Breads, pit.         1.450         2605         277         300         391         4000         475           Breads, pit.         1.450         2650         2777         306         1551         2650         408           Breads, pit.         1.450         265         2777         306         1304         2246         3799           Breads, pit.         1.456         662         1551         2650         4068           Breads, pit.         1.436         6662         1551         2656         4664</td><td>Fortals, pis./ss.         103         hits         676         6831         1076         1389           Fortals, pis./ss.         107         hits         6767         6683         6767           Fortals, pis./ss.         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18          159/18         <th cols<="" td=""><td><math display="block"> \begin{array}{c c c c c c c c c c c c c c c c c c c </math></td></th></td></td<>	Event Ar 1 (2 = 7/8 in.)           Breads, pits./is.         -155         103         kits         675         766         831         1078           Breads, pits./is.         -155         103         kits         675         766         831         1077         2268           Breads, pits./is.         -1250         275         k610         5609         6673         6672         77969           Breads, pits./is.         1.43         0.61         0.16         0.06         0.33         0.35         0.33           Breads, pits./is.         1.43         0.61         0.16         0.06         0.31         0.35         0.33           Breads, pit.         1.450         2605         277         300         391         4000         475           Breads, pit.         1.450         2650         2777         306         1551         2650         408           Breads, pit.         1.450         265         2777         306         1304         2246         3799           Breads, pit.         1.456         662         1551         2650         4068           Breads, pit.         1.436         6662         1551         2656         4664	Fortals, pis./ss.         103         hits         676         6831         1076         1389           Fortals, pis./ss.         107         hits         6767         6683         6767           Fortals, pis./ss.         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/17         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18         159/18          159/18 <th cols<="" td=""><td><math display="block"> \begin{array}{c c c c c c c c c c c c c c c c c c c </math></td></th>	<td><math display="block"> \begin{array}{c c c c c c c c c c c c c c c c c c c </math></td>	$ \begin{array}{c c c c c c c c c c c c c c c c c c c $

Table 5.4 Strain, Stress, Thrust, Moment, and Deflection; Tests B-1A, B-1B, B-2, B-3, B-4, B-5

Gage	Measurement			150	200	250	JOC	115	18/1	400	144	500
			100	<u>150</u>				315				
1	Strain, µin./in.	177	322	257	<u>بلبار</u>	15	-80					
la	Stress, pei Strein, µin./in.	1'70 138	3220 276	2570 497	1440 814	160 1173	-800 1422					
	Stress, psi	1380	2760	4970	8140	11187	11 <b>86</b> 1					
i-1 <b>a</b>	Thrust, 1b/in. Moment, in.~1b/in.	102 -0.14	194 -0.16	245 0.84	311 2.36	385 4.04	420 4.83					
\$	Strain, µin./in.	-49	-81	-65	-65	-32	-16					
4	Streas, pei Strain, µin./in.	-490 106	-810 185	-650 278	-650 385	-320 491	-160 بلبلخ					
	Streas, psi	1060	1850	2780	3850	4910	ونباح					
1-3 <b>A</b>	Thrust, 1b/in. Moment, in -1b/in.	19 0.55	¥3 بلار o	69 1.21	104 1.58	149 1.84	172 1.97					
2	Strain, win./in.	112	205	280	410	560	672					
2	Stress, psi Strein, µin./in.	1120 105	2050 134	2800 239	4100 314	5600 388	6720 448					
	Streas, pai	1050	1340	2390	3140	3860	4480					
- 28,	Thrust, lb/in. Moment, inlb/in.	71 -0.02	110 -0.25	169 -0.14	235 -0.34	308 -0.61	364 -0.79					
	Strain, µin./in.	172	310	9	587	742	846					
•	Stress, psi Strein, µin./in.	1720 40	3100 79	4490 158	5870 211	7420 264	8460 290					
	Stress, psi	400	790	1580	2110	2640	2900					
.ua	Thrust, lb/in. Moment, inlb/in.	-0.46	126 -0.81	197 -1.02	259 -1.32	327 -1.68	369 -1.95					
21	Deflection, in.	0.013	0.019	0.022	0.026	0.035	0.035					
62 -1a:3-3a	Deflection, in. Avg thrust, 1b/in.	0 ६1	0.003 114	0.006	0.006	0.009	0.009					
-ia:4-4a	Avg thrust, 1b/in.	70	ц <b>8</b>	157 183	247	267 318	296 367					
	9.	0.87	0.97	0.86	0.84	<b>7.8</b> 4	0.81					
	Chanta ita la	-		-	wat 1-13 (2		<b>0</b> 14	-				
	Ftrain, µin./in. Stress, psi	-76 -760	-152 -1520	-261 -2610	-4340 -4340	-626 -6260	-810 -8100	-945 -9450				
L	Strain, Hin./in. Streas, psi	243 ≥430	527 5270	889 8890	1256 11402	1705 12049		••				
-1 <b>a</b>	Thrust, 1b/in.	54	122	198	264	311						
	Moment, in1b/in.	1.12	2,39	4.12	5.84	7.15						
	Strein, µin./in. Stress, pei	-35	-42 -420	-49 -490	-69 -690	-97. -970	-124 -1240	-131 -1310				
•	Strain, µin./in. Stress, psi	109	212 2120	316 3160	453	515 5150	611 6110	625 5250				
34	Thrust, 1b/in.	24	55	87	125	136	158	161				
	Moment, in1b/in.	0.51	0.89	1.29	1.84	2.15	2.59	2.66				
	Strein, µin./in. Stress, psi	100 1000	207 2070	331 3310	446 4460	577 5770	700 7000	708				
L.	Strain, µin./in.	65 650	123	212	2940 2940	388 3880	465	482				
-28	Stress, pol Thrust, lb/in.	54	1230	176	240	314	379	367				
	Moment, in1b/in.	-0.12	-0.30	-0.42	-0.54	-0.67	-0.83	-0.80				
	Strain, µin./in. Stress, psi	115 1150	257 2576	379 3790	بلوبا ماوبا	629 6290	744 7440	751 7510	<sup>1</sup>			
<b>L</b>	Strain, µin./in.	41 420	116	105	247	316	384 3840	391				
48.	Stress, psi Thrust, 1b/in.	51	1160	1850 183	2470	3160 307	367	3910 371				
	Noment, in1b/in.	-0.26	-0.50	-0.68	-0.87	-1.10	-1.27	-1.27				
22	Deflection, in. Deflection, in.	0.007	0.017 0.004	0.023	0.027 0.013	0.033 0.016	0.044 0.016	0.0 <b>47</b> 0.016				
1a:3-3a 2a:4-4a	Avg thrust, 1b/in.	39	89 114	143 180	195 241	224 311	3973	390				
<b>68.34948.</b> .	Avg thrust, 1b/in.	53 0.74	0.78	0.79	0.81	0.72	<b>37</b> 3	379				v .:
-					at 3-5 (Z -							
	Strain, µin./in.	150	326	421	434	365	258		750	0	-141	-369
L	Stress, psi Strein, µin./in.	1500 72	3860	4210	4340	3650	2580		1290	**	-1410	~3690
1.	Stress, psi Thrust, 1b/in.	720 72	••	••	••		••		••	***	44 44	
ingi -	Moment, in1b/in.	-0.27	••			••	••			••		••
	Strain, µin./in.	-67	-108	-83	-67	-62	-71		-72	-79	-77	-117
	Stress, psi Strein, µin./in.	-670 81 <i>2</i>	-1080 299	-830 414	-670 503	-620 585	-710 673		-710 789	-790 908 9080	-790 1019	-1170
3a.	Stress, pel	2120	2990	108	5030 142	\$85 5850 170	6730 196		7890	9080	10190	1107
	Torust, 1b/in. Moment, in1b/in.	0.98	1.43	1.75	2.01	5. <b>38</b>	2.62		3.03	269 3.48	305 3.87	389
	Strain, µin./in.	118	190	362	3380 3380	445	545		674	777	894	1040
•	Stress, psi Strein, uin./in.	1180 36	1900	2680	3380	4450 383	5450 417		6580 515	7770	894 8940 736 7360	10400
	Stress, pel	36 360	98 980	1660	8320	383 3830	4170		5150		7360	10110
81	Thrust, lb/in Moment, in1b/in.	50 -0. <b>89</b>	-0.32	-0.34	1 <b>85</b> -0.37	-0.43	313 -0.45		<b>381</b> -0 <b>.9</b> 0	455 -0.54	530 -4.56	667 -0.10
	Strain, Min./in.	99	165	264	343 3430	475	587 5870		734	861	1082	1663
-	Stress, pei Strein, uin /in.	990	1650	26h0	3430	4750	9870 397		7340 1340	8610	10890	18031
	Stress, pai	680	1030	1760	2390 189	3200	3970		4890	569 5690 465	689 6890 346	10660
<b>ha</b> - 2	Shrust, 1b/in Moment, in15/in.	-0.11	-0.22	143 -0.31	-0.37	858 -0.55	380 -0.67		-0.86	465 -1.03	-1.88	-0.44
		0.004	0.005	0.008	0.011	0.015	0.019		0.00	0.089	0.036	6.051
	Deflection, in.											
1413-1a	Deflection, in.	0.001	0.002	0.003	0.004	0.005	0.008		0.010	0.013	0.015	0.018
51 52 - 1413-36 - 2814-48				0.003	0.00k	0.006	0.008		0.010	-		

(Continued)

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Table 5.4 (Concluded)

<b>A</b>	M.,						Pessure, pai	215 250	1.00	her	E ~~~
	Magurement	50	100	 	<u>200</u> st B-2 (Z =	<u>-50</u> 7/8 in.)	300	315 350	400	450	500
•	Strain, µin./in.	65	158	201	208	201	187	158	129	100	43
	Stress, pei Strain, µin./in.	650 22	1 <b>58</b> 0 111	2010 244	20 <b>8</b> 0 377	2010 544	1 <b>87</b> 0 710	1580 899	1290 1071	1000 1270	430 1457
-1 <b>.</b>	Stress, psi	220	1110	2440 145	3770 190	بلبلو 242	7100 292	8990 باباد	10710 390	11440 439	11941 466
-18	Thrust, 1b/in. Noment, in1b/in.	-0.15	-0.17	0.15	0.60	1.21	1.85	2.61	3.32	3.94	4.40
	Strein, µin./in.	-51 -510	-32 -320	-6 -60	6 60	13 130	6 60	o O	-12 -120	- 32 - 320	-45 -450
4	Stress, psi Strein, µin./in.	164	272	369	451	563	653	787	910	1040	1158
-34	Stress, psi Thrust, 1b/in.	1640 37	2720 76	3690 118	4510 149	5630 187	6530 214	7870 256	91u 292	104-00 3219	11149 361
•-	Moment, in1b/in.	0.76	1.07	1.32	1.57	1.94	2.28	2.77	3.25	3.77	4.21
	Strain, µin./in. Stress, psi	187 1870	315 3150	460 4600	581 5810	737 7370	862 8620	1008 10080	1152 11133	1321 11581	1923 12145
2	Strain, µin./in.	14	75	136	186 1860	254 2540	308 3080	351	455	573	1070
-28	Stress, pci Thrust, 1b/in.	140 65	750 127	1360 194	249	322	380	3510 442	521	5730 600	764
	Moment, in1b/in.	-0.61	-0.84	-1.14	-1.39	-1.70	-1.95	-2.31	-2.43 880	-2.22 1044	-0.43
	Strain, <u>µin./in</u> . Stress, psi	101 1010	191 1910	303 3030	397 3970	529 5290	633 6330	759 7590	8800	1044	1764 12075
•	Strein, µin./in. Stress, pai	34. 34-0	103 1030	168 1680	229 2290	306 3060	367 3670	443 4430	516 5160	611 6110	926 9260
-4 <b>e</b>	Thrust, lb/in.	يقها	96	153	203	271	325	391	454	538	741
<b>C</b> 1	Moment, in1b/in. Deflection, in.	-0.24 0.007	-0.31 0.009	-0.48 0.011	-0.59 0.014	-0.79 0.019	-0.94 0.023	-1.11 0.026	-1.28 0.030	-1.52 0.035	-0.82 0.043
C2	Deflection, in.	0.001	0.001	0.002	0,004	0.007	<b>ა.009</b>	0.013	0.015	0.017	0.019
-1a:3-3a -2a:4-4a	Avg thrust, 1b/in. Avg thrust, 1b/in.	33 55	83 112	132 174	170 226	215 297	253 353	300 417	341 486	384 569	414 753
	q	0.60	0.74	0.76	0.75	0.72	0.72	0.72	0.70	0.67	0.55
		_			B-3 (2 = 1						
	Strein, µin./in. Stress, psi	7 70	48 480	75 750	82 820	82 820	75 750	65 650	61 610	41 410	27 270
•	Strain, µin./in. Stress, psi	103 1030	192 1920	274 2740	378 3780	મંગ્રે દાવમાં	603 6030	722 7220	819 8190	ցեր Յրեր	1055 10550
-1 <b>a</b>	Thrust, 1b/in.	36	78	i13	149	187	220	256	286	320	352
	Noment, in1b/in.	0,34 بالباب	0.51 بالبة_	0.70 -35	1.04 -35	1.45 -41	1.86 -61	2.31 -83	2.67 -104	3.18 - <b>1</b> 30	3.62 - 148
• .	Strain, µin./in. Stress, pei	-440	-440	-350	-350	-410	-610	-830	-1040	-1300	-1480
۰.	Strain, µin./in. Stress, psi	153 1530	270 2700	364 3640	452 4 <b>52</b> 0	561 5610	670 6700	773 7730	899 8990	1033 10330	1156 11144
ač-	Thrust, 1b/in.	35	73 1.11	107	136	169 2.12	198	224 3.01	258 3-53	293 4.09	327
	Moment, in1b/in. Strein, µin./in.	0.69 158	268	365	1.71 461	574	2.57 671	768	3-23 877	992	1183
	Stress, pei	1580	2680	3650	4610 178	5740	6710	7680	8770	9920	11213
<b>1</b> .	Strein, µin./in. Stress, psi	10	69 690	121 1210	1780	243 2430	3050 305	374 3740	4460	515 5150	6790
-26	Thrust, lb/in. Noment, inlb/in.	55 +0.52	110 -0.70	158 +0.86	20 <b>8</b> -1.00	266 •1 - 17	316 -1.30	371 -1,39	430 -1.52	490 •1. <b>68</b>	612 +1.68
	Strein, µin./in.	218	371	502	625	775	897	1040	1153	1209	1179
•	Stress, pei Strein, µin./in.	2180 -32	3710 -8	5020 18	6250 58	7750 100	8970 142	10400	11136	11280	11203 <b>736</b>
_	Stress, pai	-320	-80	180	580	1000-	1420	1870	261.0	4090	7360
-42	Thrust, 1b/in. Noment, in1b/in.	60 =0.88	118 -1.33	169 -1.70	~2.00 255	284 -2.38	338 -2.66	399 -3.00	459 -3.12	-2.71	619 -1.46
<b>C</b> 1`	Deflection, in.	0.005	0.008	0.010	0.013	G-016	0.019	0.023	0.026	0.030	0.035
C2 -la:3-3a	Deflection, in. Avg thrust, lb/in.	10:004 36	0.007	0.010 110	0.011 143	0.012	0.014	0.016 240	0.019	0.021 307	ા ગ્ર≋્ય ગ્રંથણ
-2214-48	Ave thrust, 10/in.	58 0.62	114 J.67	164 0.67	215	275	327 0.64	305 0.62	1445. 0.64	506 0.61	611
	X	0.02	0.01		B-4 (2 - 2				0.04		
	Strain, win./in.	-49	-36	- 14 ° -	29	60	78	98	111	122	111
•	Stress, psi Strain, µin./in.	-490	-360 215	-40	290 399	600 506	7 <b>80</b> 610	980 735	1110	- <b>986</b>	1110
	Stress, pai	1290	2150	31:50	3990	3060	6100	7350	8540	9860	11028
-12	Thrust, 1b/in. Moment, in1b/in.	26 0.63	58 0.86	9 <b>6</b> 1.09	1.39	184	224	271	314	361	
1	Strain, win./in.	-130	-177	-194	-203	-212	-227	-252	-279	- 303	- 125
4	Stress, pei Strein, µin./in.	1 <b>300</b> 191	-1770 300	-1960 392	-2030 468	-2120 552	-2270 631	-2520 736	-8790	- 3030	- 32%) 1054
	Stress, pai	1910	3000	3510	4 <b>68</b> 0 86	5520 110	6310 131	7360	8460 184	9700 217	10%40 237
-34	Thrust, 1b/in. Moment, in1b/in.	1.13	1.68	2.06	<b>≝</b> .36	2.69	3.02	3.48	3.96	4.60	4.66
!	Streis, win./in.	174	278	370	453	555	651	765	873	1008 10080	1100
	Strein, pei Strein, pin./in.	1740 -22	2780 26	3700	4530	5550 265	6510 344	7650	8730 537	64.5	11021
- 22	Stress, pai Thrust, 10/in.	-290	260 99	970 152	1780	2650	3440 323	4400 392	5370 458	6h50 537	9460 (00/
	Moment, in1b/in.	-0.69	-0. <b>8</b> 9	-0.96	-0.97	-1.02	-1.08	-1.14	-1.18	-1.36	-0.57
•	Strein, µin./in.	219 2190	361 3610	4940 4940	617 6170	761 7610	890 8900	10h2 10h20	1189	1337	1480
4	Strein, pin./in.	- 90	-43	-17	15	7610 540	93	137	193	CB3	12
-48	Strees, rei Thrust, lb/in.	-500	-430	-170 195	150	540 265	930 319	1370	1930	2830 514	4290 505
· •	Hommat, in1b/in.	-0.95	-1,42	-1.80	-2.12	-2.49	-2.81	-3.19	-3.45	-3.36	-2.99
C)	Deflection, is. Deflection, in.	0.009 0.006	0.014	0.017	0.075	0.083	0.026	0.030 0.718	0.034	0.039	0.044
AU				~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~		V 1 U 4 T				10 × 540 3	- C 14 16 2
CH -1a13-3a -Da13-4a	Avg thrust, 1b/in. Avg thrust, 1b/in.	23 52 0.44	49 101	81. 154	113	147 266	178 321	3 <b>00</b> 314	249 453	2 <b>8</b> 9	317 626

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Table 5.5 Strain, Streas, Thrust, and Humant; Tests 5-6, 5-7, 5-8, 5-9, 5-10

0						Oversteast	WTY, DEL				
	<u>Heartment</u>	<u></u>	100	150	<b>3-6 (2 - 0</b>			350	- 430	-150	
1			~~~								
1	Strain, µin./in. Stress, psi	-239 -2390	207 2070	416 416	233 2330	-65 -650	-174 -1740	-498 -4980			
1a.	Strain, pin./in.	349	264	232	791	1385	1699	2665	6471		
1-18	Stress, pei Thrust, lo/in	3490 36	24	2320 210	7910 333	11 <b>758</b> 416	12047 648	12069	13050		
	Noment, in -1b/in.	2.07	0.13	-0.6	1.96	4.73	5.36	6.30	••		
3	Strais, pis./is.	-54	123	508	516	189	185	162	131		
34	Stress, psi Strein, µin./in.	-540 144	1230 220	2080 275	51 <b>6</b> 0	1890 508	1850 613	1620 743	1310		
	Stress, pai	1440	2200	2750	3720	5020	6130	7230	1050		
3-34	Thrust, 1b/is.	29 0.70	111 0.34	157	191	225	259	200	296		
2	Noment, in1b/in.		383	0.24	0.55	1.10	1.51	1.98	2.30		
٢	Strein, µin./in. Stress, pei	257 2570	303 3630	4940 4940	646 6460	877 8170	998 9980	1167 11172	1230 11334		
24	Strain, µin./in.	-45	័រ	108	240	336	399	463	527		
2-2	Strees, pei Thrust, 1b/in.	-450 69	130 129	1080 196	2400	3360 394	3990 454	4630 528	5270 565		
	Moment, in1b/in.	-1.06	-1.30	-1.36	-1.43	-1.90	-2.11	-2.43	-2.31		
la la	Strain, win./in.	239	301	37	446	597	729	883	880		
h-	Stress, pei Streis, µis./is.	2390 -16	3010 63	3710 158	4460 222	5970	7290	8230 459	8800 523		
	Stress, pei	-160	630	1580	2220	285	. 36ko 36ko	1990	5230		
4-44	Thrust, 1b/in.	72	118	172	217	207	355	\$17	496		
	Moment, in1b/in.	-0.90	-0.84	-0.75	-0.79	+1.10	-1.29	-1.28	-1.26		
1-1a:3-3a 2-2a:4-ba	Avg thrust, 1b/in. Avg thrust, 1b/in.	33 71	129 124	184 184	268 253	321 341	355 405	387 473	511		
	q	0.46	1.04	1.00	1.04	0.94	0.87	0.82	**		
				Test 1	-7 (2 - 7/1	6 in.)					
1	Strais, µin./in.	-145	294	641	716		558	101	148 ·	366	277
	Stress, pai	-1450	2940	6.10	7160	6410	5580	NONO	100	3060	2770
18	Strain, µin./in. Stress, pri	247 2470	175	111 1110	247 2470	455	671 6710	876	10620	1362	1605
1-18	Thrust, 1b/in.	33 1.36	152	244	313	356	399	N-3	491	547	564
	Noment, in1b/in.		-0.42	-1.87	-1.65	-0.65	0,10	1.39	2.61	2.96	3.53
3	Strein, µin./in. Stress, psi	510 51	21 <b>8</b> 0	367	149	502 5080	530	590	606	652	700
36	Strain, pin./in.	145	154	3670	218	273	5300 350	5900 393	6060 1466	6520 539	7000
	Struce, pai	1450	1540	1790	5180	2730	3500	3990	-66C	5390	5990
3-34	Thrust, ib/in. Nement, inlb/in.	94. 0.444.	121 -0.23	-0,66	217 -0.81	-0. <b>61</b>	-0.63	-0.69	<b>348</b> -0.49	-0.40	-0.36
2	Strein, sin./in.	859	339	413	562	THE	911	996	11.00	1303	1410
-	Stress, pei	2590	3320	4130	5680	7080	9110	9960	11078	11538	11827
	Strain, #in./in.	-8	50 500	136	2110	29A	394	-	531	614	701
2-24	Stress, pci Thrust, lb/in.		100	176	251	3940	. 3940 . 191	-	5310	6240	7010
	Noment, in10/in.	-0.9	-0.99	-1.00	-1.84	-1.44	-1.82	-1, 9	-2.09	-2.05	-1.75
<b>b</b> 1	Strain, pin./in.	896	310	123	586	605	893	1007	1160	1339	2 Martine
<b>h</b> a	Stress, pói Strein, µin./in.	2580 10	3100	186	5860 870	6950 354	8950	100TG	605	11612	213 <b>4</b> 4 904
-	Stress, pai	100	1090	1860	2700	3540	4370	5860	6090		986
h-ha	Thrust, 1h/1n.	·	194	198 -0.83	859	341	-1.61	i igt	574	7000	717
	Noment, in1b/in.	-0.87	-0.73		-0.90	-1.80		-1.69	-0.92	-1.71	-0.85
1-1413-34 2-2414-44	Ave thrust, 10/1m. Ave thrust, 10/1m.	- MA 85	137	2) 197	865 255	304		100 Mil	557		493 666
	•	0.92	1.06	1.13	1.0	0.91	9.80	0.79	9.75	0.7	0.78
				Surt.J	H (1 + 2/8	12.)	1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 -		· · · ·		
1	Dirain, pin./in.	-97	137 1970 851 8510 136 0.10	292 346 346 808 0.20	349 3490 445 445 845 845 845		34	356	349 3490	319	
1.	Stress, pai Strein, pin./in.	-970	1310		3690	3660	years	356	3490	3190	
	Atrest, pel	-970 261 2610 73	8510	3400		6830	368 9680 784	-		**	-
1-16	Thrust, 1h/in.	3	186	808	865	321	378	-			<b>4%</b>
	Manant, in16/18.	. Jaime	0.80	0.30	0.27	0.90		1.9	**	**	-
3	Streis, min./in. Stress, pui		777	979		1364		- 1. <b></b>	**		5 - 5 - 5 - 5 - 5 - 5 - 5 - 5 - 5 - 5 -
3	Wrote, uls./is.			-417	-	- <b>*</b> *				-	-48
1	Bereis, pei Devet, 13/1s.		925 9250 -877 -870 -870 -877	979 -417 -4170 -4170 -4170 -4170 -4190	1172 11105 -174 -174 -176 -176 -177			89		**	-6140
3-34	Manana, inlb/in.	-1.0E	-1.75	-4.98	-5.77		**	**	**		9% 9%
	Strain, sin./in.		101		550	700					
	Street, jui	8180 2180	3890	4800 8005	3390	7000	8740 8740 479	9970	11151	114.38	1.1
h, i	Birnin, sin, /in.	16 160 16	305 60 600 132 -0.85	805	559 090 1995 0905	373 3730 349	MA	997 9970 946 901 -1.79	(67	1.007 133-39 695 6950 625 -1.45	
	Stress, pal Thrust, lb/in.		132	8050 806 -0.79	2712	349	4990	901	35		
1 1	Hummit, in10/1n.	-1.72	-0.84	-0.79	-0.5	+1.15	-1.9	=1.39	1199 11151 699 990 309 -1.71	-1.1	
• · · · ·	Strain, sin./in.	175	199	130	416	7980 7980	617	THE		10240	173
•	Street, pol	1790		3300	4166 871 8710	<b>5980</b>	415				
		- <b></b>		***		3510	767		· · · · · · · · · · · · · · · · · · ·		
	Strain, pin./in. Strain, pil	170	- 1080		47.19	3944	4720		7630		····· 7160
• • •-••	Street, pel Ternet, 14/10.	170	179	199 1990 198	- #X3	306		1790		999 9995 921	1160 - <b>4</b> 5
	Stress, pai Derest, 15/18. Noment, 1813/18.	170 62 -0.56	299 2300 105 1060 119 -0.9	-0.46	-0.91	306 -0.45	4290 960 -0.83	+1.05		981 =1.99	1473 13449 7340 7340 633 -1.48
4 4-46 1-1019-96 8-8016-66	Street, pel Ternet, 14/10.	170	119 -0.94 117 116	178 -0.56 196 196	- #X3	306 -0.65		-1.05	-1.19 557		-1.45

Course & much

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Table 5.5 (Omeluded)

_									100	19	
<b>58.</b>			21.0								
					9 (2 = 1-3/	(معلي					
	Strain, pin./18.	-89	*	107	160	189	192	201	2h9	289	2
	Street, pel	-49 -890 261	340 483	3090	1840	1890	1980 888	2010	2490 1068	2090 1108	25 12
	Steels, sta./is.	200	403	495		111 1110		996 9960 377	10680	11286	14
18.	Streen, pai Servet, 15/1s.	*	1480	196	275	298	331	377	496	+76	مبر ع.ا
	Noust, inib/in.	1.8	3.44	1.37	1.61	298 1.84	2.25	2.51	2.80	3.10	3.
	Storin, sin./in.	-40	-9	52	180	172	180	215	258	<b>#03</b>	2
	Pirees, pil	-69 -690	-90	690	1800	1725	1800	8150	258 2580	2630	29
	Strain, Min./18.	205	330	900 3800	N66	531	617	696 6960	789	890	9
_	Strees, pti. Street, 15/1n.	2000	3300	3800	190	5310	6170 259	236	7890 340 1.67	8900 381	7
	Manuet, in1b/1s.	6.98	1.19	1.15	1.22	1.36	1.54	1.69	1.67	2.14	2.
		809	379	106	648	770	876	1091	1163	1346	1
	Munia, sia./ia. Stress, pri	2000	3790	4860	6680	7700	8760	10910	11168	11601	119
	Strain, sin./in.	63	198	4860 ste aliso	379	499	519	607	713	806	
	Person, pel	430 430	3790 198 1980 1980	200	3790	4590	5190	6C70 552	7130 608	9060 670	9
	Thrust, 12/1s. Meanst, in13/1s.	-0.52	-0.6	250 -0.72	-0.86	399 -1.09	453 -1.26	-1.70	-1.52	-1.29	-0
		-		-		-	885	1069	1163	1334	1 <sup>j</sup>
	Strein, sin./in.	211 2110	384	167 1670	648 6480	776 7760	8850	10490	11168	31617	ามี
	Strees, pel Strein, pin./in.	41	149	295		397	461	551	683	723	9
	Street, and	110	149 1490	255 2550 288	307	3970	461C	5510	6830	7230 947	9
•	Thread, 3h/3m.		173	-0. <b>8</b> 8	310	361 -1.33	437 -1.49	520 -1.75	579 -1.85	-1.63	-0
	Honord, inlb/in.	-0.60	-0.83		•1'WV			-1-17		-	
113-30 114-14	Ang threat, 10/1a.	51 6.60	175 181	14	883	260	295 MAS	337	385	130	1
16.24-b6	Any thrus, 13/1s.		0.66	239	319	390 0.67	0.66	536 0.63	0.57	6 <b>59</b> 0.65	0
					-10 (2 - 2-5	( <b>6.12.</b> )					
	Strain, in./1s.	-113	-11	1.51 565 865 265 265	89 890 542 5480	116	134 1340 742	1480 1480	149 1690	384 1840	1
	Stress, pti. Strein, pin./in.	-1130	- 340		. <b>3</b> 6	64	744	837	350	1065	
	Hirene, tel	199	-670 351 3510	NARD .	540	dian.	5000	8370	9500 39A	10650	11
<b>A</b>	Street, jal. Street, 15/25.		1.67	195	805	346		390	36	406	3
	Manual, in12/18.	1.10		1.0	1.59	1.85	2.1Å	8.43	2.75	3.10	3
	Strain, sin./is.	-67	482.34	-15	<u>а</u>		30 -	37	96 960 844		
1.1	Streen, pol. Strein, uin./is.			-490 397 3970	-NO	880 941	300 638 6380 813	370	ALA	670	1
	FIRES, HIR./12.			NATO	490 4500 145	5410		7080	<b>8040</b>	990	10
	Streat, st. Strut, b/ta.	0.87		: 11A	145	183	215		2.67		
	Hunne, inIb/In.	0.89	1.51	1.56	1.60	1.85	· #.12	8.36	2.47	3.03	. 3
	Strain, sin./in.	214	A11	346 9460 818	869 0868 095 0065	· · · · 764	913	2011	7800	1349	1
	Strees, pal	23.60	110	540	6960	290		10780	11257	11699	, u
	Brein, sin./in.		1340	218	250	3760	1	527	689	719	
	Strees, pel. Terest, 15/10.	n	171	0815 848	101	310	MAY	580	970 -1.09	1190	-1
	Numat, inib/in-	-0.7	-1.05	-1.19	-1- <b>2</b>	-1-37	-1.99	-1.98		-1.65	-1
	Strein, sis./is-	304	1	573	647		979	1086	1785 1248 148	1418	1
- 	Otress, pai	80h0 -	4380	5730	667	858 8580 276	4790	10660	12-07	11633	- 11
	Pursia, uin./in.	-13	- <b>36</b>		19	276	334	4100		170	
	Norone, jul. Thyradt, 22/18.	-130	161		301	8760 360 -1.96	334 3360 477	NAK -	<b></b>	616	
<b>-</b> 1	Manual, 28.+33/28.	-4.16	-1.34	5730 130 130 451 451 -1.53	198 1980 381 -1.45	-1.96	-2.86	-1. <b>3</b>	-2.9	4.8	-4
	Ang Marinet, 15/24.			135 840 0.96	175	815	-	1 <b></b>	30	10	5. 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 - 1910 -
test. In									- 200		
	Any sarres, 12/1a.	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		and in	2.40	3 <b>16</b> 0.59	0.57	905	72 <b>8</b> 5 <b>9.96</b>	0.56	0

				Table 5.6					
is,	Stress,	thrust,	Honest,	and Defluction;	Tests C-1,	c-2,	c-3,	C-4,	C-5

L		<u> </u>		_7_	_2	100				- 20	300	159	400	-10	900
	Strain, µin./in.	159	634		••	Tast C-	(2 . 0	<u>18.</u> ]							
	Stress, pei Strein, µin./in.	1990	6340 156	'n	-453										
	Stress, pai Ingust, lb/in.	1700 36	1560 87	710	-4530										
	Nument, in1b/in.	0.00	-0.19	••	••										
	Strein, pin./in. Mrees, pei	262 <b>868</b> 0	470 4700	616 6160	739 7390										
	Strain, µin./in. Strain, pei	-69 -690	-106 -1060	-119 -1190	-125 -1250										
	Thrust, lb/in.	21 -0.13	40	55	-0.35										
	Noment, in1b/in. Strain, min./in.	-0.13	~0.23k	-0.30 334											
	Stress, pri Strein, min./in.	1250	2340	3340 205	4500										
	Stress, pai Threat, lh/in.	480 19	1210	2050 59	3200										
	Monant, in1b/in.	-0.03	-0.05	-0.05	-0.35										
	Mrain, uin./in. Peres, pei	159 1590	273 2730	387 3870	470 4700										
	Strein, sin./in. Stress, pai	24	69 620	110	153 1530										
	Threat, 1b/in.	20	37	55	69										
	Moment, islb/is. Deflection, is.	-0.05	-0.09	-0.11 +0.0 <b>08</b>	-0.13										
-3a	Deficition, in. Ang thrust, ih/in.	0.003 29	0.003	0.003	0.009										
	Avg thrust, 1b/in.	20	36	57	त्त										
		1.45	1.68	••	**	Dest C-4	( <b>7 - 1</b> /1	6 <u>1a.</u> )							
	Breis, sis./is.		591			1987	1604	1902							
	Stress, pri Strein, pin./in.		1910 185			10970	160h0 495	19090							
	Strees, pil Darwei, 15/1n-		1850			3000	6990 231	8000							
	Herent, in1b/in.		-0.12			-0.31	-0.15	-0.44							
	Strain, uin-/in. Strees, pai		3580			5190 519	666	7600							
	Strein, sin./in.		-179			-159	-80	20 100							
	Thevet, 12/1n. Houset, in15/1n.		19 -0.21			-1590 40 -0.27	67 -0.31	86 -0.30							
	Strain, sin./in.		155			m	186	64							
	Mress, psi Mrein, uin./in.		1950			3110	4860 790	6480 1044							
•	Stress, pel Deust, 11/18-		2430			5960 98	7900 140	10440							
	Honort, in16/1a.		0.0A			0.09	0.15	9.34			- - -		•		
	Birela, pia./in. Wrose, poi		203			540 1940	817	1085			· · ·				۰
	Birula, ula./ia. Mirusa, pii					145	750	100%							
	Brost, 14/18.		56			-0.0	173 -0.03	-0.08							
· .	Barnet, in -15/10.		0.00h		·	9.90	0.00	0.013	•						
-	Periorian, in.		0.000			-0.001	-0.091	-0.001					an an taon Taon tao amin'		
	day thrust, 10/10.		· · · 51.			97 163 0.94	196 196	193 209 0.98		a tag	1				
			¥- <b>3</b> 4			3mm 0-5	12 . 5/								
	Mpula, sis./is.		-		на 19 1	981	1994		16.39		ADES CADES		145	833	891
	Mania, pil./18.		1120	на 1. с.		911 920 185 186 186 180 190	S TRUES		1439 14390 601 6010 946 -0.14		1319		2485 24250 2810 28100	#3390 #1000 #1000 576 0.01	BTI 1
	Steven, phi Thrunt, 25/10. Meanst, 1015/10.	ana Geologia	1150				3940		6010 346	344	1319 13190 377 -0.38	17010	- 10	27000 376	**
н N.,			-0.12	e La de la c		-0.30	. <b></b>		4.14	-0.39	-0.3	-0.21	-A.09	0.07	**
	Persin, sis./is. Straws, pol Persin, sis./is.		1				PALIN ALL		7544	6610		973	1706	1000	13
1.1	Pirela, nia./ia. Pirelo, poi		4 S W 2 3 5			12884 B	FERN P		144 510 525 525 525 525 525 525 525 525 525 52				1190	1394 1394 1394 13940 13940 13940 13940 13940	19 19 19 19 19 19 19 19 19 19 19 19 19 1
	Street, phi Threat, 15/18. Heavet, 1815/18.		11 0				-0.12		-0,QL	9.01	÷.05	817 9.01	0.05	20.04	31 0.0
	Annia, sia./ia.					524	771 7710		1046	1300	LONT	1949		2045	
	- Blance and		250	an a		- 53MO- 300	7710		10460 776	1,000		19430	100		04 84
	Direis, sis./is. Diress, pil Direst, 15/1s. Namet, 1615/1s.		<b>e</b> 6632				330 3300 143		1046 1046 776 776 100 40.11	1388 13880 1094 1094 1094 1094 1094 1094	1007 10070 1001 1001 1001 1001 1001				-
	Names, 18-13/18-		-9.03			-4.66	-1.10		-0.11	-0.11	-0.12	+D.13		-9.18	
	Stania, sin./3a.	en di seri Seri	119	11 A.	a W		474 1940 1911 1910	4. 1 - 1 1. 1	695 6930 515 515 5150 149		1137			1977	
	Plevala, sia./ia. Suras, sii Barut: 20/10.	1	2						515			1105 11050 210	11000	1983	17
	Revel: 20/10. Result, 1013/10.		119 1190 40 40 40	· · ·	et i pi	877 205 205 205 205 205 205	-0.2		-0.04	-9.07		4.13	1000 1000 1000 1000 1000 1000 1000	1977 19770 1982 1983 1985 1985 1985	2.234
	Suffection, in.		-0.00 0.000 30 1.39			0.000 0.000 11 1.45	0.000		406.0 0.000 0.000 0.000 0.000 0.000 0.000	0.000	0.013 9.005	0.019	6.00	6.051 0.013 131 151	.0.0
							-0.000		3.000	0.000	0.00	0.000	0.000		
)- <b>3</b> 4	Buflerties, is.		0.001				-0-001 340 315 2.50			8)1 81		114 114	0.000 381 393	LUI	0.01

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## Table 5.6 (Concluded)

_				144	124	Overgreesure. pdi 195 200	250	300	350	400	450	500
<u></u>	_Herrist	<u> </u>	<u> </u>	100			<u>- 670</u>				<u>-77</u>	
				Test C-	2 (Z =	7/16 in.)						
	Strain, µia./ia.	426		694	886	1034	1120	1180	1193	1217	1266	1290
	Stress, pai	4260		6940	8880	10340	11200	11800	11930	12170	12660	12900
	Strain, win./in.	14		98 980	265	460	711	1018	1408	1798	2245	2717
	Stress, psi	140			2650	4600	7110	10180	14080	17980	22450	27170
18	Thrust, 1b/in.	46		87 0	127	164	201 -0.16	242 -0.07	286 0.09	332 0.23	386 0.39	(بلبئ 0.5€
	Hennet, in1b/ia.	-0.17		-0.24	-0.25	-0.23	-0.10	-0.07	0.09	0.43	0.39	0.90
	Strain, win./in.	31		107	176	267	351	443	550	642	179	909
	Stress, pai	310		1070 480	1760	2670 801	3510 964	4430 1154	5500 1369	6420 1566	7790 1811	9090 2032
	Strain, µin./in.	302 3020		4800	647 6470	8010	9640	11540	13690	15660	18110	20320
34	Stress, p41 Thrust, 1b/1n.	37		65	91	117	145	176	211	243	265	324
, <b>m</b>	Moment, in1b/in.	0.11		0.15	0.19	0.22	0.25	0.29	0.33	0.37	0.42	0.45
	Annala with the	83		158	253	372	515	669	847	1052	1332	1640
	Strein, µin./in. Strees, pei	830		1580	2530	3720	5150	6690	8470	10520	13320	16400
	Strain, pin./in.	75		133	195	270	373	489	616	763	982	1200
•	Stress, pel	750		1330	1950	2700	3730	4890	6160	7630	9820	12000
- 24	Thrust, 1b/18.	17		32	49	71. -0.04	98 -0.06	127 -0.07	161 -0.09	200 -0.12	255 -0.14	312 -0.18
	Moment, in1b/in.	0.0		-0.01	-0.02	-0.04	-0.00	-0.07	•0.09		-0.14	
	Strain, pin./in.	100		177	269	380	507	645	799	971	1207	<u>յիրի</u> Հիրին
	Strees, pei	1000		1770	2690	3800	5070	6450	7990 1104	9710 1312	12070 1581	1ներիՕ 18940
1	Strain, µin./in.	118		243	390	5եթեն Տերթեր	726 7260	909 9090	11040	13120	15810	18400
h-	Strees, pai	1190		2430 46	3900 72	102	136	171	209	251	307	361
-lee	Thrust, lb/in. Moment, inlb/in.	0.01		0.03	0.05	0.07	0.09	0.11	0.12	0.14	0.15	0.16
				0.003	0.006	0.011	0.015	0.021	0.027	0.031	0.037	0.044
C1 C2	Deflection, in.	0.002 0.001		0.003	0.000	0.003	0.005	0.007	0.010	0.011	0.013	0.015
-lat3-34	Deflection, in. Avg thrust, 1b/in.	42		76	109	141	173	209	249	266	336	383
2.4-4	Ave thrust, 1b/in.	51		39	61	87	117	149	185	226	261	337
	۹. · · · ·	2.00		2.00	1.79	1.62	1.48	1.40	1.35	1.27	1.20	1.14
				Test C	- <u>) (Z -</u>	7/8 in.)						
	Strein, pin./in.	243		503	694	902	1058	1180	1386	1562	1701	1857
	Stres, pti	2630		5030	694 6940		10580	11800	13880	15620	17010	18570
۵.	Strain, pin./in.	196		413	708	925	1181	1457	1673	1915	2209	SPOR SPOR
	Stress, pet	1960		4130	7080		11810 246	14570 290	16730 337	19150 382	22090 430	465
•1 <u>8</u>	Thenest, lb/in. Homest, inlb/in.	48 -0.02		-0.04	154 0.01	201 0.01	0.05	0.11	0.11	0.14	0.20	0.22
							-					
	Strein, sin./in.	863		1654	2257			**	••		••	
	Piress, pei	8630 -445		16540 -891	22570 -1270		-1907	-2317				
•	Persia, pia./ia. Eurosa, pei	-4450		-8910	-12700		-19870	-23170	••		**	••
-34	Thrust, 1h/in.	46		84	109			**		•-	**	**
· •	Nummet, in13/in.	-0.53		-1.03	-1.42	••	~ *	••		. <b>**</b>		
	Strain, min./in.	223		416	616	820	1046	1298	1582	1839		**
	Heres, pai	2830		4160	6160		10460	12980	15820	18390		
· ·	Strain, Min./In.	57		188	352		733 7330	969	1837	1479	1771	2034
	Struce, 301	570		3860	،»منذ	5210	7330	9690	12370	14790 365	17710	20340
-86	Derest, lh/in. Homest, inlh/in.	-0.07		-0.09	106 -0.11		196 -0.13	-0.13	310 -0,14	-0.15	••	••
	and the second s			•				-				
	Mreis, sin./in.	839		433	611		10130	1237	1493 14930	17230	1976	22)1 22)7
	Stress, ptl	2390		1330	6110		961		- oft	1190	1437	166
<b>6</b>	Strain, nin./in. Strans, pri	· •			2260	3770	3610	750	9840	11900	14370	1667
-	Thrust, 1b/15.			960 58	98	130	17	219	515	320	375	14 M
	Hannah, in12/18.	-0.09		-0.14	-0.16		-0.18	-0.19	-0-81	-0.21	-0 <b>.22</b>	-0.8
<b>C</b> 1	Deflection, 18.	+0.005		-0.011	-0.016	-0.017	-0.019	-0.017	-0.015	-0.013	-0.010	-0.000
	Deflection, 18.	0.003		0.003	0.003	0,0 <b>0</b> h	0.006	0.008	0.010	0.012	0.014	0.011
-1413-34	Ang thrust, 1h/13.	47		98	131	120	185	234	291	343	* **	
	Ave threat, 13/15.	1.6	1	1.48	. 99			2 <b>2</b> 3 4			**	
	1 <b>4</b> 1				1.32	4.						

				Table	5.7					
raia,	Stress,	Thrust,	and	Nonest;	Tests	¢-6,	C-7,	c-8,	C-9,	C-10

tt.

	Man average A		20		100	154			1. <u>26</u>	240	2/24	350	100		900
	Magazetanat	_2	<u> </u>	<u>B</u> _	_100	<u>125</u>	<u> </u>	. <u></u>	<u></u>	<u> </u>				<u> </u>	
1	Strain, sin./is.	-508	-836	-312	555	1443	<u>6 (2 = 0</u> 2033	<u>ليھد</u> 2459							
la	Stress, psi Strein, µin./in.	-5080 709	-8360 876	-3120 1312	5550 1106	14430	20330 718	24590 332							
1-1a	Stress, psi. Thrust, 1b/in.	7090	8760	13120	11060	10100	7180	3320 307							
	Homest, in -1b/in.	0.49	0.69	0.66	0.22	-0.17	-0.53	-0.86							
3	Strain, µin./in. Stress, psi	9 90	74 740	259 2590	435 4350	602 6080	741 7410	615 6150							
34	Strain, µin./in. Stress, psi	163 1630	<u>336</u> 0 کار	523 5230	718 7180	887 8870	980 9800	1017 101 <b>7</b> 0							
3-34	Thrust, lb/in. Moment, in1b/in.	17 3.05	45 0.11	86 0.11	1 <b>27</b> 0.11	164 0.11	1 <b>89</b> 0.10	202 0.08							
2	Strain, win./in.	297 2970	451 4510	613 6130	809 8090	943 9430	1184 11840	1249 12490							
28	Stress, psi Strain, µin./in. Stress, psi	127 127	264	396 396	515 5150	673 6730	7% 7%	873 8730							
2-2	Thrust, ib/in.	47	79 -0.08	111 -0.09	146	179 -0.11	217	233 -0.15							
4	Noment, in-1b/in. Strain, win./in.	208	333	448	567	672	761	838							
4.	Stress, psi Strein, uin./in.	20 <b>8</b> . 136	3330 293	430	5670 518	6720 660	7610 767	8380 850							
ليسابع	Stress, pai Thrust, 1b/in.	1360 38	.3 <b>990</b> 69	4300 . 97	51 <b>8</b> 0 119	6600 147	<b>767</b> 0 172	8500 126							
	Moment, in1b/in.	-0.03 20	-ə.o. 25	-0.01 98	-0.02 155	0.00 217	0.00 246	0.00 255							
1-1413-34 2-2414-44	Avg thrust, 1b/in. Avg thrust, 1b/in.	43	74	104	- 133	163	195	210							
	4	0.47	0. <b>3</b> 4	©.94	1.17	1.33	1.26	1-21 No (1-1)							
1	Strain, win./in.		-218		1037	THAT C-	<u>7 (2 - 3</u> / 2202	10 1A. J	1240	3549	3767	420A			
- 1a	Stress, psi Strein, uin./in.		-2180 738		10370 237		22020 -223		32400 -251	35490 167	37554 530	38935 599			
1-14	Strees, pui Thrust, lb/in.		738 7380 57		2370 140		-2230 218		-2510	1670	5300 473	5990 524			
	Moment, in1b/in.		0.39		-0.32		-0.98		+1.41	-1.36	-1.31	+1.41			
3	Strain, µin./in. Stress, psi		809 8090	,	1518 15180		50900 5090		2520 25200	2019	3088			-	de la composición de la compos
3 <b>n</b>	Strain, Min./in. Stress, pei		-419 -4190		-723 -7230		-1113 -11130		-1417	-1633 0(£01-	-1735	-1908			
3-34	Thrust, lh/im. Moment, imlh/im.		43 -0.50		67 -0.90		107 -1.29		-1.59	134 -1.81	-1.92				
5	Strein, Min./ia. Strees, psi		303 3030		640 6400		6200 6200		866 • <b>NG</b> 0	1185	1372	2626			
26	Strain, pin./in.		3810		100		7%0 7300		201	1337	1547 15470	-1814			
2-2	Stress, psi Thrust, 16/10. Moment, in.+16/in.		-0.01		108		145 0.04		20	2/7	<u>्रथा</u> च.पर	310			
<b>b</b> <sup>2</sup>	Strein, pin./in.		306		559		77		1073	144.9	Litt				
48	Stress, pai Strein, uim./im.		3060		5590		7760	ar an Na San A	10190		1576	Source 1973			e de la composition
4.4.3	Stress, pai Thrust, 11/18.		201-0 65		67 <b>90</b> 116		1960		04401 215	13890	399	1110			
	Monent, in-th/in-		-0.01		-0.03 114		-0.0 <b>1</b> 163	•	90.90 600	-0.03 279	-0.85 				
i-in13-30 9-2019-60	Avg thrust, 15/1s. Avg thrust, 15/1s.		6		109	с	197	÷		295	340	694			an a
	al an		0.77		1.09		A (3 + 5	64 IL 1	1. 3 JUNE	NA COMPANY					
	Brain. min./in.		-11		171				1073	1999	nas	2005			
1.	Strees, pei Strein, sin./in.		-120		9790 998 9980		1543 19430 641		ALL DO	1	21090 1719	2239	11714 (14.1)	3915	
1-18	Street, pt. Dyunt, 11/18-		9700	n na st	9986 173		6146 645				17100	20395	313	29140 178	-
	Rammi, inth/in.		39		-9.13		-9. <b>)4</b> .		100, 100 100	10.20 10	-0,16	112.00 ·	0.65 1861	0.05	9.48 Unit
<b>.</b>	Strain, sin./in. Strace, pei	•	57		199					SCHO	10090	1134	- 185	1.3	1400
34	Strain, sin./in. Strand, pti	· · ·	310 310		1990 636 6360 109		814 Blint		-		141	1376	1994		1790
3-34	Therest, 1b/in. Humont, 181b/in.		0.16		0.11		15A 0-09		196	0 Q. 💭	247 0 10	0.10	911) 911 9	330 0.13	364 10.14
2	Strein, pis./in- Otress, psi		300 3000		542 5420		8100		HORA C			1751			Pho:
26	Strain, pla:/in-		A HE		991 991				1093		1005		2113 81130	2174 4374	
2- <b>in</b>	Struce, ps. Shrust, by Ls.		70		145		144	1.1	236	<b>30</b>	-0.0		4.4	12	793 -0,3
- <b>6</b> 12	Hennit, in19/in-				-				1273	19406		2107			1961
	Mrees, joi Strain, min./in.						114 111)		11400		1141 1141				JUSIL
anda .	Stress, pei Torest, 10/18.		Midő Wi		8090		11100			11,000	23415 199	547	196 5.05		Safa Safa Safa Safa Safa Safa Safa Safa
	Honnet, in16/16.		0.02		0. <b>0</b> 4		0.07 198		0.58 250	4.05 	9. <b>ch</b>			<b>2.</b>	
لللأخل: ه∶مة هامه الع: ماة	Arg thrust, 14/18.		2	1. 1	142 145		17				1)* 111 10.81		335	0.17	*
	•		Ø. 🗰		Q. 🙀		\$: <b>#</b>		. V. 73	<b>~~</b>		4.40			0.3

( same)

## Table 5.7 (Concluded)

11.2

in the later of the

het					0 175	200	10	300	10	400	350	. 50
				Best Call				<u> </u>				
	Strain, pip./in.			1		1603	1 <b>85</b> 4	2019	2187	2382	2549	26
	Meuc, pai			, 		16090	10540	20190	21870	23820	- 200	267
	Mersia, wis./is.	54	66	i li	A	1206	1527	1992	2314	2679	بقورته	31
	Strain, pai	100	63	5 10 <b>1</b>		12060	15270	19980	23140	26790	29180	318
le.	Throut, 1h/1a.	71	17		355	318	372	641	495	557	601	6
	House, in -th/in.	0.19	e 0.0	) -Q.	.09	-0.13	+0.13	-0.0i	0.05	0.12	0.15	0.3
	Strein, pic./in.	ي. مەدر				983	1127	272	1300	1532	1691	18
	Marana, pal	1240			60	9034	14270	12720	13000	15320	01601	180
	Mrain, pin./in.	254 2940				896	1013	1219	1410	1601	1718	10
	Marust, 12/18.				67	8460 201	10130	.2190 214	14100	16010	171.80	100
	Human is 13/1s.	0.0				-0.04	432 44.05	-0.02	0.01	0.03	- 375 C. 01	a 0.
	tireis, pis./is.	601		а <i>и</i>		1360	1758	~~ <b>?</b> ®	2509	294.	3342	л
	Street, pel	NOLO				13600	17080	20020	25090	29410	33420	37)
	Strain, sin /in	473			<b>A</b> 1	1 2:34	1637	2066	312	2008	3158	35
	Ptrees, pat	1730			00		16370	2066	24120	26020	11683	35
<b>h</b> .	Therape, 1h/1m.			5 <sup>–</sup> i	č 94	4	14	457	544	632	716	
	itement, in-it/in-	0.03	-0.0	)	<b>\$</b> 77	42.W.	-4 (3	-0.03	-0.0k	+0.05	-0.07	, «ĝ
	Bosta, sta./is.	398	) tê		NS.	:15:	1449	ieta .	2116	Acc		Э
	Strene, sal	3980			lar-	1 <b>8</b> 10	$T_{p} \sim 0$	:633:	21160		260-10	26
	Mrs.a, sis/is	816 	2	·	لغا	1242	1.1	619	1579	2380	20	2
	Streas, pat	716C			ion i	10120	27.30	162.90	10790	21851	ALC: YEL	24
•	Threat, ih/in. Humant, inlh/in.	-0.07				241. -0.01	298) 	<b>380</b>	-0.10	000°	542 40, 11	)
					7 <b>7</b> 6				-0.10	- 09	-0.11	••)
123-30 Al-614	Ang threat, ib/ib.	5			ta a	-6j		350	1404 .	431	-	`
figerit.	Any Units, 34/12.	0.73			La de la companya de La companya de la comp	269	191	419 0.85	990 10.85	569 © 79	639	0
	•	****		, <b>.</b>	<b>3</b>	2 <b>79</b>	C. 44	v. sg	197 - 19 <b>1</b> 8	N 17	W . PP	. <del>4</del>
				THE CALL (2	+ 7.7 14:)						•	
	Pereis, sis./is.		57	۶ e	<b>*</b>	1154	37	1560	1737	1060	2181	2
	Sizes, jal	100				11310	13760	15600	17570	1969	180	23
	Renia, ala /1a.	50				1592	183-	2014	8.9	2543	2777	*
	- Strengt, and	SUE	1			15550	10330	<b>BUIND</b>	21195	25430	27.19	- 30
	Barnet, 1h/La.	. 56	1 . W					198	-	499	543	
· ·	Names, in -12/1a.	17 ×4	9.1	<u>9</u> .	3 <u>1</u>	20	0.1	11	3.0	0. <b>A</b>	ų, <b>m</b>	\$
	Breis, sis /is.	104			**		847	-	1.144	1272	136	
	Meresa, pat	101-			982	7580	-	2970	114.00	1710	1300	- 45
	Brain, sin /in.	<b>X</b> **				រូនដុំតិ	. 12%	100	1613	1799	1987	<u>,</u>
	Maten, pel	م <b>دينو</b>			<b>1</b>	202 🗮	1.1	240,000	16110	7.00		4
۱. ۱	Burnet, Sh/La. Report 18-33/1a.	4.5			1	. AU	378 0-17	345	303	75		
					3	- <del>2</del> .₩	Q 17	6 IT	9.19	Q. 84	Ø. 🏔	÷
	Strain, als /35	-			<b>**</b> - 1 - 1	114	1741		2997	1715	1276	5
1.1	Paress, pai	1400				: \$5#	1.20	199,54	6397×	mx	1000	13
	Mrain, ais /in Mrain, pai				8¥	119) 1190	1500			25-4 2-200		1
1. 	Revel, HVLA					1.1799 1.179	35	410			677% 654	19
•••••••	Annust, 1411/18.					-4 - 4 <b>1</b>	- 2 <b>a</b>	-5	-2.00	- 2 🖉	4	÷Č
	dienta, sta /ta					0.0	.TIT		210		293	
a sura	R.7016, 263						171.75		19.00		275	
	Persia, sla./13	1 <b>4</b>			4						67%	1
	Reves, set	41	17.0			10.00			1993	- <b>#</b> ***		
<b>.</b> . 1	Brut, IN/ta.	<b></b>		1		1	<b>345</b>	14.3	* 🗱	<u> 1</u>	500	
	Renat, in -th/im.	-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1	±	k	🖷 💡 🖓	-1. M	÷.	*	-1. <b>3</b>	×2.13	-2.17	<b>1</b>
						1 <b>21</b>	246	14.9	178	113	-	3
***	Ang Statute, 12/12	27	14									
122	Ang signal, 15/12			È la la de				13.4 C 71	1414 171 - 1	*		1

		Table 5.8									
Strain,	Stress,	Thrust,	Honest,	<b>1</b>	Deflection:	lests	8-1,	8-2,	8- J		

Jaco	Head and the second	50	100	150	198		WYL WYL I		- SC	350	100 H
<u> </u>			100		113	200	20-	<u> </u>		170	
	Strain, uin./in.	604	1.3413	2995	<b>hest <u>8-3 (8</u></b> 4193	+ C 1n.)	11000				
	Stress, psi	<b>4903</b>	1361 6308	7527	8111		10760				
	Strain, uin./in. Streat, bai	- 397 - 3970	-620 -4945	-9:4 -5767	-1179 -6123						
-2	Thrust, 10/18.	62	140	2. 6.8	266		••				
	Hommet, islb/is. Strain, µis./is.	36.36 94a	-169	•.e	5.05 -296		-480				
	Street, pol	-960	-1690	- 980	-2080		-4579				
	Strain, sin./is. Stress, psi	3700	761 5314	1536 6487	·						
•	Thrust, 10/10. Hommant, in10/10.	7C -1.42	167 -2.64	263 42.79	••		**				
	Birais, uis./is.		70	239	320		560				
	Stress, pul	-470 264	700	2390	3200		4736				
	Strain, pin./in. Strung, pai	2640	346 3460	4450	522		691 5131				
	Thrust, lb/in. Humant, inlb/in.	7 <u>1</u> -1.09	135 -0.97	-0.73	267 -0.56		322 -0.12				
	Strain, sin./in.	.51	139	221	370		444 -				
	Stream, pai Strain, pin./in.	510 97	1390	2210 209	3700		3440 358				
	Stress, pol	97.0	1560	YORC	2200		-580				
	Thrust, 16/15. Monument, is16/18.	-0.16	966 	140 9.34	194 0.50		22 <del>9</del> 9.67				
	Strain, Min./in.	200	*(1)	491	600		58-0				
	Stress, joi Strein, uis./is.	2800 -206	-010	-202	4893 -189		505¢ -176				
	Stread, pai	-2050	-2200	-208-7	-1890		-1.60			• 1	
0	Thrust, 16/14- Homent, is16/14-	24 1.71	59 2-19	93 2.42	127 2.58		2.60				
	Strain, sie /in.	120	271	399	517		5 14				
	Strees, Sai. Strein, uin./in.	1200 16	2710 10	5996 59	34675 64		4367 63				
	Streap, pil	<b>1</b> .0	5:00	7 <b>%</b> -	then.		680				
	Thrust, 14/1a. Honest, 1a14/1a	- 55 . e. 12	144 9.59	1.25	1.53		1.60			· .	
	Persis, with fin	æ	*	172	172		76E -				
	BLPROD, DEL		:	2710 606	99 <b>9</b> 8 - 7 <b>9</b> 6		5397 92 -				
	Birais, sis. As. Biras, sei	4.740	· 741	は読	3474		2115				
•	Struct, 14/1e Manuel, 1413/14	144	- 18 - 19	1 <sup>8</sup> 9 -6.5.	-0.29		\$59 -0.14		1997 - N. 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 199 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 -		
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•	Arnie, più Arnie, 210-520		• 30%	1.1	1413		4 360				
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A Lin	Through the status	轉之	1.94 	23) (1.22	£19 -41 - 37		- <b>₩</b>				· · · · ·
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• 14	dafte film, in.	7 (363) 447	-	1. ar - 1. 141	195 195		**	• • •			
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					the first it						
	Birtis, sir. 's. Street, pet	14 k 14 k	191	145	<sup>-</sup>		n an	Same	4998年 時期間		
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.:	Street. DE	<b>€</b>	14	-		190		19		-	
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	Strain, str./in. Strain, ybi	-34 -340	**					175	995 B	福井	577 <b>0</b>
1.1.4	Planter, and the	278. 1788	1988 1998			770	8 - E 1		27 M		43.4% WNK
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	Reports las-lasta		64- <b>2</b>					1254			
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	Waters, Waters		1.5	<i>514</i>		274		887	-	195	***
	Similar in the	8.94 .	ş.#	1. (1. (1. (1. (1. (1. (1. (1. (1. (1. (		š.,		\$.5£	2.78	ē.#	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1
	Minister, with Paris Minister, 2013			-		907 Shirth	e e est		100 M	* (14) * (14)	1957 1963
	Withing all .756-	47.53	-1.95	-161	•				1	190	794. 3-9993
•	Parist, 24/20		-1790	1.39		100			<b>*53</b>	1999 1999	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1
	Minini, M. Altyle	1.46	1- <b>9</b>			2.82	· .	1.9	1. <b>1. 19</b>		÷.45
	Strain, site./16 Strain, site	1416 1416				905 - 1443 -		1.3%e - 51009		192 194	
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Table	5.8	(Contluded)

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	Manurement	30	100	150	0vergressure, 175 200 205	250	300	350	400	
		<u></u>						-		تنب
					z = 7/16 in.) (Continued)					
	Strain, win./in.	-92	-12	161	518	1313	2788	433. 816-2	5668	
	Stress, pei Strein, uin /in.	-920	-120 480	1610 716	4678 1091	6259 1995	7409 3690	52(9	0734 6512	
	Stress, pri	327 9270	4579	5496	6033	6955	7875	8538	3082	
14	Thrust, 15/in.	75	152	254	352	429	497	543	579	
	Nombut, in1b/in.	-1.48	-1.72	-1.28	-0.51	-0.24	-0.16	-0.13	-0.12	
	min, win./in.	34	69	165	k 3k	1220	2701	3821	5385	
	Suress. pei	0 مالا	690	1650	4340	6164	7360	7940	3615	
	Strair, min./in.	173	302	590	3811	2318	4306	6163	7953	
	Stress, pai	1730	3020	4967	6061	7141	8159	8938	9680	
16	Thrust, 15/10.	0.10	121 -0.82	2 <sub>5</sub> 4 1.22-	348 -0.61	436 -0.36	्र06 -0.26	-0.35	595 -0.33	
	Moment, in1b/in.	-0.49								
	Deflection, in.	0.017	0.026	0.035 C.927	0.043	0.055	0.070	0.086	0.105 0.041	
9-10	Deflection, in. Avg thrust	0.017	0.022 90	157	0.031 231	0.035 310	0.037 381	0.039 431	484	
13-14	Avg thrust	75	149	262	354	433	490	542	580	
-3 •	9	0.55	0.60	0.60	0.65	0.72	0.77	0.80	0.83	
	•			<b>7</b>	<b>E-1 (2 = 7/8 in.)</b>					
	Consta uta lla	-65	636		1237	1626	1090	2520	2253	34
	Strain, µiz./in. Streas, pai	365 3650	635 ,4984	917 5723	6185	6579	1989 6949	2530 7262	3253 7656	7
	Strain, µin./in.	-226	-272	-266	-214	-98	29	324	753	i
	Stress, psi	-2260	-2720	-2650	-21: 2	-960	290	3240	5293	6
	Thrust, 1b/in.	45	109	167	228	292	340	400	442	
	Ecment, in1b/in.	2 <b>.06</b>	2.94	3.11	2.93	a - 37	1.85	1.07	0.74	3
	Strain, win./in.	37	70	148	397	967	1821	2933	3828	4
	Streas, pai	370	700 307	1460	3870	<b>5906</b> 1935	6777 2 <b>8</b> 99	7492 4179	7944 5528	8
	Strain, uin./in. Stress, psi	151 1510	307C	531 4712	937 2775	65+4	7472	8105	8672	9
	Thrust, 1b/in.	61	123	217	327	417	465	507	540	
	Moment, in1b/in.	-0.40	-0.83	-1.23	-0.56	-0.35	-0.23	-0.22	-0.25	-0
	Strain, uin./in.	-57	24	190	538	1304	2129	3236	4429	5
	Stress, pai	-570	240	1900	473C	6250	7034	7648	A210	é
	Strain, µin./th.	273	439	682	1051	5058	3369	4866	6176	7
	Stress, pei	2730	4390	5107	6033	6976	7714	8394	8943	9
	Thrust, 1b/in.	70	150 -1.46	257 -1.15	354 -0.49	430 -0,26	480 -0.24	522 -0.26	558 -0.26	-0
	Moment, in1b/in.	-1.16			-					
	Str+in, µin./in.	304	456	609	913	1217	1522	2282	2891 7468	3. T
	Stress, psi Strain, µin./in.	3040 26	4516 75	4916 178	5712 387	6161 827	6472 1111	7121 2202	3070	3
	Stress, pei	260	750	1780	3870	5487	6053	7075	7565	ว้
	Thrust, 15/in.	107	173	242	325	364	407	461	489	
	Noment, in1b/in.	0.98	1.34	1.17	0.54	0.23	0.15	6.02	-0.03	-0
	Strain, µin./in.	112	115	112	162	349	686	1129	1629	1
	Stress, pai	1120	1120	1120	1650	3490	5118	6072	6582	6
	Strain, µin. in.	56	169	294	450	662	918	1238	1433	1
<b>`</b>	Stress, pei	560	1690 91	2940 132	4500 199	5055 294	5725 352	6183 398	6362 421	ť
)	Thrust, 1b/in. Noment, in1b/in.	55 0.20	-0.50	-0.64	-1.01	-0.49	-0.21	-0.04	0 07	C
			187	288	475	893		2059	2808	. 3
	Strein, µin./in. Stress, pei	72 720	1870	2880	475	5660	1339 6286	6994	2000 7421	1
	Strain, µin./in.	52	124	195	357	685	1047	1690	2390	
	Stress, pai	520	1240	1950	3570	5115	5988	6644	7182	7
2	Thrust, lb/in.	40	101	157	269	350	399	հեր	475	
	Moment, in10/in.	0.07	0.22	0.33	0.38	0.19	0.10	0.13	0.08	C
	Strain, µin./in.	-33	49	262	687	2021	2781	4057	5464	6
	Stress, psi Strein uin /ir	-330 28 <b>3</b>	۷90 455	2620 718	5121 1201	6972 2431	7405 3674	8054 5088	8645 6501	9 7
	Strain, µin./in. Stress, psi	2810	477	5202	5145	7206	7867	8487	9078	ģ
14	Thrust, 1b/in.	81	164	281	372	461	497	538	576	-
	Moment, in1b/in.	-1.11	-1.43	-0.86	-0.38	-0.08	-0.16	-0.15	-0.15	-(
	Strain, µin./in.	28	69	152	388	1053	1953	2964	4141	4
	Stress, pai	280	690	1520	3880	5994	6912	7509	8089	ε
	Strain, µin./in.	156	341	611	1081	2048	3114	4579	5987	7
	Stress, psi	1560	3410	4922	6023	6987	7587	8273 514	8865	9
16	Thrust, lb/in. Moment, inlb/in.	-0.45	133 -0.96	234 -1.28	339 -0.66	423 -0.36	472	514 -0.27	-0.27	(
	Deflection, in.	0.014	0.021	0.027	0.033	0.042	0.050	0.060	0.070	. 0.
:9-10	Deflection, in. Avg thrust	0.013 50	0.017 100	0,020 150	0.023	0.025	0.1.36 346	0.02; 399	0.029 431	· 0.
	Avg thrust	76	157	269	363	293 1446	489	06.	567	
:13-14							0.71	0.75		. 0

and the second

5.6

Gage	Messurement	Test E-6 Z = 0 in. P = 254 psi so	Test E-5 Z = 7/16 in. P = 262 psi 	Test E-4 2 ≈ 7/8 in. P = 264 ps: so =
L	Strain, µin./in.	5841	7899	5698
	Stress, psi	8803	9658	8743
2	Strain, µin./in.	8637	4881	3849
	Stress, psi	9933	8400	7955
2	Thrust, lb/in.	610	587	543
	Moment, inlb/in.	-0.40	C.44	0.27
<b>}</b>	Strain, µin./in.	3207	3949	3326
	Stress, psi	7634	8004	7693
	Strain, µin./in.	5131	6189	5344
	Stress, psi	8505	8948	8594
-4	Thrust, lb/in.	526	551	531
	Moment, inlb/in.	-0.30	-0.33	-9.31
;	Strain, µin./in.	41 <i>9</i> 7	5582	4802
	Stress, psi	8113	8694	8367
5	Strain, µin./in.	4654	4946	5392
	Stress, psi	8305	8427	8615
5-6	Thrust, lb/in.	534	556	552
	Moment, inlb/in.	-0.07	0.09	-0.09
7	Strain, μin./in.	5878	7562	6045
	Stress, psi	8819	9518	8889
3	Strain, µin./in.	3944	59 <b>0</b> 4	4565
	Stress, psi	8002	8830	8267
7-8	Thrust, lb/in.	547	596	558
	Moment, inlb/in.	0.29	0.24	0.22
)	Strain, μin./in.	5098	6552	5300
	Stress, psi	8491	<b>9099</b>	8576
10	Strain, µin./in.	3622	4991	3653
	Stress, psi	7841	8446	7856
9-10	Thrust, lb/in.	532	570	535
	Moment, in.+lb/in.	0.23	0.23	0.25
u	Strain, µin./in.	4415	6537	4503
	Stress, psi	8204	9093	8241
12	Strain, µin./in.	5 <b>3</b> 47	5148	4890
	Stress, psi	8596	8512	8404
11-12	Thrust, 1b/in.	546	572	541
	Moment, in1b/in.	-0.14	0.20	-0.06
13	Strain, µin./in.	3233	5350	3757
	Stress, psi	7647	8597	7909
14	Strain, µin./in.	3949	4969	4297
	Stress, psi	8004	8437	8155
13-14	Thrust, lb/in.	509	554	522
	Moment, inlb/in.	-0.13	0.06	-0.09
15	Strain, µin./in.	2651	3581	3630
	Stress, psi	7331	7821	7845
16	Strain, µin./in.	4898	6172	5499
	Stress, psi	8407	8941	8660
15-16	Thrust, lb/in.	914	546	537
	Moment, inlb/in.	-0.38	-0.39	-0.28
1-2:9-10 5-6:13-14	Avg thrust Avg thrust q	571 521 1.10	579 555 1.04	539 537 1.00

Table 5.9

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				T	able 5:10						
strain.	Stress,	Thrust.	Moment,	and	Deflection;	Tests	D-1,	<b>D-2</b> ,	D-3,	p.4,	D-5

Gage	Measurement	25	50	<u>?5</u>	<u>90 95</u>	100	130	150	175	180	190
				Te	st D-1 (Z = 0 in.)						
	Strain, µin./in.	293	684	2881	16194						
	Stress, pai	2930 -184	4249 -1534	625) _004	-1036						
	Strain, µin./in. Stress, psi	-1640	-3677	-4898	-1036						
2	Thrust, 15/in.	35	58	179							
	Moment, in10/1n.	163	3.23	4.23	+- 1000						
	Strain, µin./in. Stress, psi	-57 -570	12 120	27 270	-1710 -5512						
	Strain, µin./in.	195	224	<b>#02</b>	7109						
	Stress, psi Thruct, lb/in.	1950 45	2240 77	3583 137	8157 278						
έ <b>μ</b>	Moment, in1b/in.	-0.89	-0.75	-1.25	-4.61						
	Strain, µin./in.	190	-59	-542	-398						
	Stress, psi Strain, µin./in.	1900 -54	-990 255	-3957 1303	-3571 2142						
	Stress, psi	-540	2650	5161	5845						
6	Thrust, 1b/in.	44, 0.86	67 -1.14	125 -3.61	223 +2.89						
	Moment, ir1b/in.	88	-1.14 -32	-3.01	-2.03						
	Strain, µin./in. Stress, psi	380	-320	-120	2680						
	Strain, µin./in.	-39	112 1120	236 2360	81 810						
i	Stress, psi Thrust, lb/in.	-390 16	1/50	73	113						
	Moment, in1b/in.	0.45	-0.51	-0.87	0.66						
	Strain, µin./in.	-136	341	1609	3040						
	Stress, psí Strain, µin./in.	-1360 163	3403 -222	5424 -1073	6380 -1718						
	Stress, pai	1630	-5550	-4963	-5517						
.0	Thrust, lb/in. Moment, inlb/in.	9 -1.05	39 1.98	67 4.60	108 5.23						
	Strain, µin./in.	-1.07	-6	-66	90						
	Stress, psi	600	-60	-660	900						
	Strain, µin./in.	-11 -110	81 810	237 2370	188 1880						
-12	Stress, psi Thrust, 1b/in.	16	24	56	90						
	Moment, in1b/in.	0.25	-0.31	-1.07	-0.35						
	Strain, µin./in.	525	389	204	642						
	Stress, psi Strain, uin./in.	3922	3544	2040	4162						
	Stress, pai			••							
14	Thrust, lb/in. Moment, inlb/in.										
	Strain, µin./in.	-348	-321	-417	-3980						
	Stress, psi	-3424	-3210	-3627	-6864						
	Strain, µin./in. Streas, psi	799 4484	1065 4956	1725 5525	9744 9171						
16	Thrust, 10/in.	102	148	191	220						
	Moment, in1b/in.	-2.97	-2.88	-3.08	-6.43						
L 2	Deflection, in. Deflection, in.	0 -0.002	0.018 0.024	0.078 0. <b>956</b>	0.165						
2:9-10	Avg thrust	- 22	49	123							
:13-14	Avg thrust 9	0.50	0.73	0.98							
			L.P.C		t D-3 (Z = 7/16 in.)	4,043	14400				
	Strain, µin./in. Stress, psi	-11 -110	476 3799	1585 5403		6888	10634				
	Strain, µin./in.	195	-251	-832		-1530	-9231				
2	Stress, psi Thrust, lb/in.	1950 60	-2510 67	-4554 102		-5356 182	-8974 140				
•	Noment, in1b/in.	-0.73	2.39	4.22		4.84	8.54				
	Strain, µin./in.	159	235	320		353	-738				
	Stress, psi Strain, µin./in.	1590 -26	2350 56	3200 82		3438 256	-4360 4665				
	Stress, psi	-260	560	820		2560	7156				
4	Thrust, 1b/in.	43	95	131 0.84		198 0.33	287 -3.30				
	Moment, in1b/in.	0.65	0.63	-476		-740	-3.30 -266				
	Strain, µin./in. Stress, pei	24 ( 2440	-78 -780	-476		-4364	-2650				
	Strain, µin./in.	-147	356	1260		2035	2410				
6	Stress, psi Thrust, 1b/in.	-1470 32	3W\$7 90	5124 134		5781 158	6606 261				
	Moment, in1b/in.	1.38	-1.52	-3.42		-3.92	-2.35				
	Strain, µin./in.						**				
	Stress, pai	341	407	445		428	422				
	Strain, µin./in. Stress, psi	3403	3597	3708		3659	3641				
8	Thrust, 1b/in.		••				**				
•	Moment, in1b/in.	**		 619			1490				
	Strain, µin./in. Stress, pai	74 740	285 2850	618 4113		1014 4912	5.121				
-		42	-114	-350		-506	-174				
	Strain, µin./in.		-1140	-3429 68		-3882 96	-4022 139				
)	Stress, pai	420	~~								
0		.38 0.11	56 1.40	2.91		3.53	3.65				
0 -10	Stress, pai Thrust, 1b/in.	. <u>38</u> 0.11	1.40 153	2.91 392		717	1195				
0 -10 1	Stress, psi Thrust, lb/in. Moment, inlb/in. Strain, µin./in. Stress, psi	38 0.11 13 130	1.40 153 1530	2.91 392 3553		717 4317	1195 5068				
0 -10 1	Stress, psi Thrust, lb/in. Moment, inlb/in. Strain, µin./in.	. <u>38</u> 0.11	1.40 153	2.91 392		717	1195				

(Continued)

Table 5.10 (Continued)

a	Magn	- 26	60	72	60	and the local division of the local division	pressure, p 100	130	150	175	180	190
Onge	Nonstrement	25	50		<u>_90</u>	<u>_95</u>		<u></u>			100	-170
		<b>A</b>	- ( -		z = 7/16 1	n.) (Coniir		-97				
	Strain, µln./in Stress, psi	87 870	-261 -2610	-639 -4156			-881 -4635	-387 -3538				
	Strain, µin /in.	21	618	1895			3023	4009 6873				
-14	Stress, psi Turust, 1b/in.	210 35	4113 95	5670 163			6372 202	306				
-	Moment, inib/in.	0.23	-2.56	-3.71			-3.96	-2.51				
	Strain, pin./in.	7	106	173			233	-1091 -4978				
	Stress, psi Strain, µin./in.	70 147	1060 199	1730 199			2330 387	بلبلبار				
	Stress, psi	1470	1990	1990			3538	7491 278				
-16	Thrust, 1b/in. Moment, in -1b/in.	50 ~0.49	99 -0.33	121 -0.09			198 -0.46	-3.89				
1	Deflection, in.	-0.005	0.017	0.054	2		0.100	0.167				
2	Deflection, in.	-0.009	0.015	0.047	;		0.072	0.088				
2: <b>9-1</b> 0 6:13-14	Avg thrust Avg thr st	يلايلا بلار	62 93	149			139 180	282				
-	q	1.38	0.67	0.57			0.77	0.50				
				Tes	t D-3 (2 =	7/3 in.)						
	Strain, µin./in.	32.9	2376	7863	13973							
	Stress, psi Strain, µin./in.	3190 -264	5986 -1390	8ii47 -4063	10502 -9730							
	Stresz, psi	~2640	-5235	-6897	-9166			5				
2	Thrust, lb/in. Moment, inlb/in.	18 2.05	96 4.95	159 6.57	115 8.66							
	Strain, uin./in.	-40	-222	-386	-1052		-9530					
	Stress, psi	-490	-2220	-3535	-14945		-9089					
	Strain, µin./in.	167 1670	593 4062	1493 5324	3068 6395		17656					
	Stress, psi Thrust, lb/in.	38	102	178	183							
	Moment, in1b/in.	-0.76	-2.40	-3.04	-4.30							
	Strain, µin./in.	30 300	-438 -3688	-651 -4181	->33 -39 <b>3</b> 8		166 1660					
	Stress, psi Strain, µin./in.	118	-3000	1447	1554		1206					
	Stress, psi	1180 48	4562 84	5284 120	5376 153		5077 265					
5	Thrust, lb/in. Moment, inlb/in.	-0.31	-3.30	-3.86	-3.52		-0.98					
	Strein, µin./in.	-136	للبا	515	946		1511					
	Stress, pai	-1360	440	3901	4789		5081					
	Strain, µin./in. Stress, psi											
	Thrust, 1b/in.											
	Moment, in10/in.						841					
	Strain, µin./in. Stress, psi	271 2710	527 3926	703 4288	802 11492		4572					
	Strain, µin./in.	-261	-522	-057	-657		-687					
0	Stress, pri Thrust, lb/in.	-2610 3	-3915 1	-4193	-4193 28		-4255 29					•
-	Moment, in1b/in.	1.87	3.26	3.69	3.77		3.84					
	Strain, µin./in.	-161	194	871	1419		1504					
	Stress, psi Strein, µin./in.	-1610 161	1940 -50	հ63հ _հ48	5260 -589		5333 -606					
	Stress, pei	1610	-500	-3717	-4053		-4068					
12	Thrust, lb/in. Moment, inlb/in.	0 -1.13	47 0.86	87 3-34	128 3.71		134 3.74				•	
	Strain, µin./in.	QÂ	-230	-325	-168		691					
	Stress, psi	940	-2300	-3250	-1680		4263					
	Strain, µin./in. Stress, psi	-240	478 3805	918 4731	1083 4971		762 4410					
14	Thrust, 1b/in.	23	74	127	188		282					
	Moment, in 1b/in.	0.42	-2.32	-2.89	-2.24		-0.05					
	Strain, µin./in. Stress, psi	-154 -1540	-308 -3080	-411 -3609	-1101 -49 <b>8</b> 7		-7581 -8339					
	Strain, µin./in.	295	707	1521	3278		14530					
-16	Stress, psi Thrust, 1b/in	2950 46	4296 98	5348 175	6502 188		10674 195					
- 20	Moment, in1b/in.	-1.58	-2.79	-3.12	4.34		-8.03					
L	Deflection, in.	0.013	0.058	0.142	0.171		0.186					
2	Deflection, in. Avg thrust	0.007	0.040 49	0.073 84	0.088 72		0.093					
2:9-10 5:13-14	Avg thrust	36	79	124	171		274					
	q	0.31	0.62	0.68	0.42							
				<u>Tee</u>	D-4 (Z =	1-3/4 in.)						
	Strain, µin./in.		561				2896		5370			9051
	Streas, pai		3996 -348				6298 -1201		7459 •15 <b>8</b> 4			8905 -1966
	Strein, µin./in. Stress, psi		-3423				-5073		-5402			-5731
2	Thrust, 1b/in.		57				154 4.60		232 4.68			312 4.74
	Moment, in1b/in.		2.00				211		737			756
	Strain, µiu./in. Stress, psi		1430				2110		4358			4391
	strain, pin./in.		132				448 3717		1453 5290			508 733
4	Stress, pel Thrust, 15/in.		1320 89				203		320			40
•	Noment, in1b/in.		0.04				-0.58		-0.3			-0.9
	Otrain, uin./in-		-185				-1826		-2434			-238
	Stress, pel Strein, µin./in.		-1850 431				-5610 3553		-60R0 6360			
	m no marts 10 200 1/ 200 1		3667				66-2		7869			868
	Stress, pai											
5 : <b>-6</b>			-2.08				119 -5.27		203 -5.51			277 -5.2

## Table 5.10 (Concluded)

8	Strain, µin./in. Stress, psi Strain, µin./in. Stress, psi Thrust, 1b/in. Nument, in1b/in. Stress, psi	25 50 -2533 -2530 -2530 -2530 -2530 	Test D-4 (Z = 1-3/4	250	826			
	Stress, psi Strein, µin./in. Stress, psi Thrust, 1b/in. Moment, inlb/in. Strain, µin./in.	-2530 354 3441		250				
	Strain, µin./in. Stress, psi Thrust, lb/in. Moment, inlb/in. Strain, µin./in.	354						150
	Stress, psi Thrust, lb/in. Moment, inld/in. Strain, µin./in.	~ 3441		2500 127	4542 236			533 76
	Noment, in10/in. Strain, µin./in.	51		1270	2360			440
		-2.13		123 0.43	248 0.63			32 0.3
	Stress, psi	318		1346	2366			315
.O	Strain, µin./in.	3180 -263		5198 -665	5980 •673			643 -45
.V	Stress, psi	-2630		4210	-4226			-375
	Thrust, lb/in. Moment, inlb/in.	18 2.05		106 3.91	192 3.66			26 2.8
	Strain, µin./in.	478		1410	2397			312
	Stress, psi Strain, µin./in.	3805 -252		5253 ~697	5998 -692			642 -46
	Stress, pai	-2520		-4216	-4266			-377
12	Trust, 1b/in. Moment, in1b/in.	67 2.40		107 3.97	191 3.70			26
	Strain, uin./in.	-230		-1270	-1577			-129
	Stress, psi	-2300 476		-5132 3913	-5396 6811			-515 934
	Strain, µin./in. Stress, psi	3799		6826	3045			901
14	Thrust, 1b/in. Moment, in1b/in.	7% -2.32		201 -)	279 -4,47			36 -3•8
	Strain, uin./in.	-2.32 -175		-223	811			-5.0
	Stress, pai	-1750		.230	4511			500
	Strain, µin./in. Stress, ps:	եկե 3706		360 3459	985 4869			354 663
.5	Thrust, 1b/in.	83		269	305			38
	Moment, inlb/in. Deflection, in.	-2.06		-0.46 	-0.13			-0.5
	Deflection, in.	0.023		0.080	0.093			0.10
9-10 13-14	Avg thrust Avg thrust	38 76		130 165	162 244			28 31
	q	0.50		0.79	0.66			0.9
			· · · · · · ·					
	Strain and An	1001	<u>Test D-5 (Z =</u>		11000			
	Strain, µin./in. Stress, psi	1591 5408		7368 8257	11022 9587			
	Strain, µin./in. Stress, pai	-871 -4634		_4137 -6929	-5661 -7584	<b>-990</b> 6 -9234		
	Thrust, 1b/in.	97		139	180			
	Moment, in1b/in.	4.29		6.63	7.28			
	Strain, μin./in. Stress, psi	-397 -3567		-521 -3913	-715 -4313	-2740 -6204	-6975 -8105	
	Strain, µin./in. Stress, psi	972 4842		2104 5822	4472 7073	10259 9350	20912	
	Thrust, 1b/in.	116		199	263	296		
	Moment, in1b/in.	-3-18		-3.31	-3.28	-5.45		
	Strain, µin./in. Stress, psi	-201 -2010		-331 -3310	-77 -770	602 4080	878 4649	
	Strain, µin./in. Stress, pai	361 3462		998 4896	1617 5431	1669 5475	1408	
	Thrust, 1b/in.	52		137	250	320	5251 325	
	Moment, in1b/in.	-1.97		-2.92	-1.79	-0.46	-0.19	
	Strain, µin./in. Stress, psi	198 1980		529 3930	541 3954	660 4200	680 4241	
	Strain, µin./in.	-194		-289	89	194	206	
	Stress, pai Thrust. 1b/in.	-1940 1		-2890 69	890 184	1940 226	2060 230	
	Noment, in1b/in.	1.38		2.60	1.12	0.71	0.67	
	Strein, µin./in. Stress, psi	468 3776		1404 5247	2635 6141	3160 6442	3160 6442	
	Strain, µin./in.	- 381		-1026	-1251	-1269	-1273	
	Stress, psi Thrust, lb/in.	-3520 24		-4922 51	-5116 131	-5131 162	-5135 162	
	Moment, in1b/in.	2.85		4.52	4.73	4.65	4.65	
	Strain, µin./in. Stress, psi	-350		368 3482	703 4268	855 4601	876 4645	
	Strain, win./in.	127		8	208	389	408	
2	Streas, psi Thrust, lb/in	1,270 بلار		80 122	2080 233	3544 267	3600 270	
	Moment, in1b/in.	-0.52		1.25	0.67	0,36	0.35	
	Strain, µin./in.	-127 -2170		-233	0	710	1356	
	Stress, pei Strein, µin./in.	304		-2330 1000	0 1588	4 303 1795	5206 1726	
14	Stress, pei Thrust, 15/in.	304:0 58		4900 162	5406	5584	5524	
• •	Moment, in1b/in.	•1*95 20		-2.51	261 -1.52	330 -0.41	349 -0.11	
	Strein, µin./in.	-228		-250	-323	-1688	-5159	
	Stream, pel Strain, µin./in.	-2280 558		-2500 1 144	-3230 3173	-5492 7639	-7368	
~	Streas, pai	3989		5196	6448	8361	10742	
.6	Thrust, 1b/in. Moment, in1b/in.	93 -2.40		-195 ⊷2 -5 ±	285 -2,42	294 -4.52	287 +6,95	
	Deflection, in.	0.038		0.149	0.176	0.184	⊷0.99 0.186	
	Deflection, in.	0.032		c , OFR	0.098	0.099	0.101	
19-10 13-14	Avg thrust Avg thrust	55 1.11		95 150	156 256	325	337	

Table	5.11
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Strain, Stress, Thrust, and Moment; Tests D-6, D-7, D-8, D-9, D-10

Gage	Measurement	Test D-10 Z = 7/8 in. P <sub>so</sub> = 97*psi	Test D-8 Z = 7/8 in. P = 116*psi so	Test D-9 Z = 7/8 in. P = 148 psi so	Test D-6 Z = 1-3/4 in. P = 160* psi	Test D-7 Z = 1-3/4 in. P = 180 psi so
1	Strain, µin./in. Stress, psi		3361 6541	17296	13714 10421	19059
2	Strain, µin./in. Stress, psi		-891 -4675	-15970	-5512 -7520	-10134 -9312
1-2	Thrust, lb/in. Moment, inlb/in.		218 3.90		251 7.16	
3	Strain, µin./in. Stress, psi		343 3409	-14665 -1071.6	-5063 -7327	-7055 -8136
4	Strain, µin./in. Stress, psi		 	5836 7659	14 <b>38</b> 0 10 <b>628</b>	19105 
3-4	Thrust, lb/in. Moment, inlb/in.			-7.30	<b>283</b> -6.90	
5	Strain, µin./in. Stress, psi		1174 5-50	9087 8918	3 <b>88</b> 1 6809	<b>538</b> 4 <b>7</b> 465
6	Strain, µin./in. Stress, psi		550 39 <b>7</b> 3	17760	-665 -4210	-686 -4253
5-6	Thrust, lb/in. Moment, inlb/in.		<b>298</b> 0.40		268 3.26	314 3.10
7	Strain, µin./in. Stress, psi		-53 -530	1468 5302	860 4612	-2511 -6067
8	Strain, µin./in. Stress, psi		1933 5702	-411 -3609	591 4057	6892 8073
7-8	Thrust, lb/in. Moment, inlb/in.		271 -1.68	170 3.14	282 0.20	216 -5.52
9	Strain, µin./in. Stress, psi	1551 5374	3195 6459		<b>28</b> 65 62 <b>7</b> 9	-1910 -5 <b>68</b> 3
10	Strain, µin./in. Stress, psi	-1306 -5163	-576 -4027		720 4 <b>323</b>	5327 7441
9-10	Thrust, lb/in. Moment, inlb/in.	29 4.83	250 3.18		358 0.62	204 -5.10
11	Strain, µin./in. Stress, psi	349 3426	-754 -4393		<b>88</b> 0 4653	508 3886
12	Strain, µin./in. Stress, psi	0 0	3827 6782		3477 6603	295 2950
11-12	Thrust, lb/in. Moment, inlb/in.	113 1.23	255 -3.49		377 -0.64	231 0 <b>.28</b>
13	Strain, µin./in. Stress, psi	517 3905	<b>665</b> 4210		20 <b>7</b> 9 <b>58</b> 07	3998 6869
14	Strain, µin./in. Stress, psi	0	480 3811	••	1935 5704	206 2060
13-14	Thrust, lb/in. Moment, inlb/in.	154 1.47	<b>261</b> 0,14		374 0.04	362 1.18
15	Strain, µin./in. Stress, psi	-2171 - <b>5863</b>	823 4535		-768 -4422	-11447 -9719
16	Strain, µin./in. Stress, psi	8019 8507	1105 4990	••	<b>59</b> 76 7720	22539
15-16	Thrust, 1b/in. Moment, in1b/in.	272 -5.06	312 -0.17		323 -3.21	••
1-2:9-10 5-6:13-14	Avg thrust		234 280 0.84	••	305 321 0.95	338

\* No failure.

.







 $M_c = 0.25 p_a R^2$ 



a. TALBOT (1908)

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b. CAIN (1929)







c. SPANGLER (1938)

d. WHITE (1960)

Fig. 2.1 Concepts of Load Distribution

102 Pso VERTICAL AXIS CROWN QUARTER POINTS 45 θ HORIZONTAL AXIS 90 SPRING LINE SPRINGING 270° MIDDLE SURFACE INTRADOS EXTRADOS INVERT 180" y Fig. 3.1 Cylindrical Shell and Ring Notation



Fig. 3.4 Nomuniform Lond Fig. 3.5 Elastic Supports Fig. 3.6 Elastic Medium





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a. Static Bonnet (2-ft Diameter, 500 psi)



b. Dynamic Bonnet



c. Spacer Ring and Posttest Diaphragm Configuration



d. Cylinder in Position

Fig. 4.2 University of Illinois Blast Load Generator





Fig. 4.4 WES Small Blast Load Generator (SBLG) Facility



Fig. 4.5 Illinois Instrum Intation Equipment

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Fig. 4.6 WES Large Instrumentation Room

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Fig. 4.7 WES Small Blast Load Generator (SBLG) Instrumentation













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Fig. 5.4 Stress, Thrust, Moment, and Deflection, Test A-3A (2 = 7/8 in.)

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DIAMETER CHANGE DECREASE PLOTTED POSITIVE INCREASE PLOTTED POSITIVE GAGE DC 1 DC 2 ¥ 010 DIAMETER CHANCE, IN 0.05 0 1.0 MOMENT, IN -LB/IN. ٥ ~1.0 -2.0L 500 f OFFSET YELD <u>Symeol.</u> O A D V CROSS-SECTION i-là 2-28 3-38 4-48 400 EMST. YELQ THRUST, LB/M. 300 -200 = DC I 100 £ 9714901. 0 4 4 5 6 7 7 7 7 10,000 GAGE ٠ 18 2 24 3 34 4 7.90 5000 STRESS, PS 210 NOO SUNFACE OVER



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Fig. 5.7 Stress, Thrust, and Moment, Test A-10 (Z = 0 in.)







Fig. 5.9 Stress, Thrust, and Moment, Test A-8 ( $Z = \frac{16}{16}$  in.)

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Fig. 5.11 Stress, Thrust, and Moment, Test A-6 (Z = 1-3/4 in.)

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Fig. 5.12 Stress, Thrust, Moment, and Deflection, Test B-1A (Z = O in.)
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Fig. 5.13 Stress, Thrust, Moment, and Deflection, Test B-1B (Z = O in.)







Fig. 5.15 Stress, Thrust, Moment, and Deflection, Test B-2 (Z = 7/8 in.)

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Fig. 5.16 Stress, Thrust, Moment, and Deflection, Test B-3 (Z = 1-3/4 in.)

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Fig. 5.18 Stress, Thrust, and Moment, Test B-6 (Z = O in.)

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Fig. 5.19 Stress, Thrust, and Moment, Test B-7 (Z = 7/16 in.)







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Fig. 5.24 Stress, Thrust, Noment, and Deflection, Test C-4 (Z = 3/16 in.)



Fig. 5.25 Stress, Thrust, Noment, and Deflection, Test C-5 (2 = 5/16 in.)



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Fig. 5.29 Stress, Thrust, and Moment, Test C-7 (2 = 3/16 ir.)







Fig. 5.31 Stress, Thrust, and Moment, Test C-9 (Z = 7/16 in.)

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Fig. 5.33 Thrust, Moment, and Deflection, Test E-3 (Z = 0 in.)

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Fig. 5.34 Thrust, Moment, and Deflection, Test E-2 (Z = 7/16 in.)

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Fig. 5.35 Thrust, Moment, and Deflection, Test E-1 (Z = 7/8 in.)



Fig. 5.36 Strain, Test E-5 (Z = 7/16 in.) and Test E-4 (Z = 7/8 in.); Surface Overpressure = 250 psi



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Fig. 5.37 Thrust and Moment, Tests E-6 (2 = 0 in.), E-5 (2 = 7/16 in.), and E-4 (2 = 7/8 in.); Surface Overpressure = 250 psi



Fig. 5.38 Stress, Thrust, Moment, and Deflection, Test D-1 (Z = O in.)



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Fig. 5.39 Stress, Thrust, Moment, and Deflection, Test D-2 (Z = 7/16 in.)

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Fig. 5.40 Stress, Thrust, Moment, and Deflection, Test D-3 (Z = 7/8 in.)

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Fig. 5.41 Stress, Thrust, Moment, and Deflection, Test D-4 (2 = 1-3/4 in.)

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Fig. 5.42 Stress, Thrust, Moment, and Deflection, Test D-5 (Z = 2-5/8 in.)





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Fig. 5.44 Cylinders of Groups A, B, and C after Tests



Fig. 5.45 Cylinders of Groups D and E after Tests



Fig. 5.46 Cylinders D-6 and D-10 after Test







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Fig. 6.1 A Group: Average Spring Line Thrusts, Crown Moments, Vertical Diameter Changes, and q Values


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Fig. 6.2 B Group: Average Spring Line Thrusts, Crown Moments, Vertical Diameter Changes, and q Values









Fig. 6.4 Peak Diameter Changes and Deflection Stiffnesses

 
 STATIC TEST
 E

 C-1
 0

 C-3
 3/46

 C-2
 7/46

 C-3
 7/46

 D-1
 0

 D-2
 7/46

 D-3
 7/46

 D-5
 2-5/46

 E-3
 0

 E-2
 7/46

 E-1
 7/46
 STATIC TEST A-1 A-5 A-2 A-3A A-4 B-1A B-1A B-5 B-2 B-3 B-4 <u>Synelot</u> 0 ... ... ... ... ... LINE. 0 3/16 7/16 7/8 1-3/4 0 7/16 7/8 1-3/4 2-5/8 AR = N<sub>y</sub> PloR 1.4 1.3 1.2 MASSIVE OR NEGATIVE 1.1 1.0 ACTIVE ) OR } 0.9 D RATIO, AN 0.7 STATIC 0.0 0.5 c-0-Š 0.4 0.3 0.2 0.1 NCE OVER STATIC SING Fig. 6.5 Static Arching Ratio

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Fig. 6.8 Relation Between Nondimensional Pressure and Equation 3.16

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Fig. 6.9 Relation Between Nondimensional Pressure and Equation 3.8

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Fig. 6.10 Relation Between Nondimensional Pressure and Equation 3.17

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#### APPENDIX A. PROPERTIES OF ALUMINUM TUBE MATERIAL

The cylindrical test specimens were cut from 12-ft lengths of Alcoa, drawn, aluminum tubing which was commercially available. Static, mechanical properties for the material are published in the manufacturer's literature, Aluminum Company of America (1960, p 59, and 1962, p 162). However, the values given are either minimum or average values, and hence do not adequately describe a given piece of tubing. Additionally, it was necessary to know the full stress-strain curve for the material up to the maximum strains experienced during the cylinder tests. In many cases, these strains far exceeded the indicated yield values.

Longitudinal tension test specimens were cut from each end and from the center section of the 12-ft lengths of tubing. The specimens averaged about 10 in. in length and were proportioned in accordance with ASTM Designation: E8-61T, <u>ASTM STANDARDS 1961</u>, Part 3 (pp 165-181).

The flat grips of the tension test machine proved unable to hold the slightly curved test specimen adequately once yielding commenced. Therefore, a special adapter was designed to accommodate the curvature of the specimen to the flat test grips.

Specimen from the tubes designated A, B, and C were all tested at the University of Illinois in a Tinius Olsen Testing Machine. It was used as a constant strain-rate device. An average crosshead speed of 0.05 in./min was used. It was first thought that the strain could be recorded adequately by monitoring with a manually operated Baldwin strain indicator. This proved satisfactory only for strains below first yield. The strain indicator operator was not able to keep a continuous balance

above yield due to the large strain changes. Hence, the system finally established utilized a Moseley X-Y plotter to record both load and strain simultaneously.

Specimens from the tubes designated D and E were tested at WES in a 30,000-lb, Riehle universal testing machine. An X-Y plotter was again employed.

Average stress-strain curves were developed for each 12-ft tube. They are plotted in Fig. A.1 and reduced to a finite number of digitized points in the tables shown on the figure. These points were used to describe the curve for the computer program.

The tension tests revealed no systematic variation in stressstrain characteristics along the length of the 12-ft tubes. The modulus of elasticity for the material,  $10 \times 10^6$  psi ( $\pm 5\%$ ), was verified by all of the tests. However, the inelastic stress-strain curves for the 6061-0 A, D, and E material varied from the average by  $\pm 10$  percent. The overall accuracy of the measurements, procedure, and reduction of data "or the 6061-T6 and 5052-0 material was within  $\pm 5$  percent of the average.

Although tubes A, D, and E were made of the same material, 6061-0, the inelastic stress-strain properties were sufficiently different from tube to tube that separate stress-strain curves were utilized in the data reduction.

Handbook yield values taken from Aluminum Company of America (1960, p 59) point up the fact that all the tubing does, in fact, exceed the manufacturer's indicated strengths. The values are indicated in Fig. A.1.

The stress-strain properties in Fig. A.1 were used in the

computation of thrust and moment under both static and dynamic loading. It was assumed that the static stress-strain relation would be a good approximation of the dynamic stress-strain relation. Aluminum is not, in general, strain-rate sensitive according to Steidel and Makerov (1960) and

Smith (1963).



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Fig. A.1 Aluminum Stress-Strain Properties

# APPENDIX B, PROPERTIES OF SANGAMON RIVER AND COOK'S BAYOU SANDS

#### B.1 Placement Techniques

Since special effort was made to place and control the quality of the soil medium, it is probably denser and more uniform than that which could be obtained in a field installation.

#### B.1.1 Sangamon River Sand

This sand was stored in closed 55-gal drums near the testing device. A 2-gal water bucket was filled with sand, weighed on Toledo scales (0.1-1b graduations), carried to the test device and sprinkled into place. The sand was placed in 6-in. lifts. After 6 in. of sand had been placed, the lift was vibrated with a probe-type, concrete vibrator (Viber Co., Model II). The probe was vibrated completely through the 6-in. lift and was positioned on 2-in. centers in an ever-decreasing spiral around the center. This process was repeated until the test device was filled (four lifts) and screeded off.

A trench was then scooped out of the center of the sand and the cylinder placed at its intended depth and leveled. The sand was backfilled in the vicinity of the cylinder in 3/4-in. lifts by sprinkling the lift in and then rodding with a small ruler and tamping with a piece of wood.

The weight of sand displaced by the cylinder placement and subsequent backfilling was measured for each test. By assuming an effective volume of soil to be disturbed during placement, it was possible to calculate the average density of the sand in the immediate vicinity of the cylinders. The calculations indicated an average density of  $105.4 \pm 1.5$ pcf. The horisontal stiffness calculations in Section 6.2 also verify the

fact that the sand was very stiff. Penetration tests were not run because of the likelihood of disturbing the cylinder specimen. Additionally, recent research at WES\* on the use of penetration tests in dense sand has indicated that inherent scatter in hand-operated penetration test data within a layer is so great that variations in density on the order of 1 or 2 pcf are effectively masked.

The overall density was established by dividing the measured weight of the sand placed by the known volume of the test device (less the cylinder volume). The overall density was very reproducible and averaged 104.0 pcf with a minimum of 103.5 pcf and a maximum of 104.5 pcf.

The strain gages were continuously monitored during the placement. The A and B groups of cylinders were insensitive to the placement, but great care had to be exercised in backfilling around the very thin C group. In all cases the tendency was for elongation of the vertical axis. However, impressed strains were kept below 50 µin./in.

B.1.2 Cook's Bayou Sand

This sand was stored in piles on the floor and shoveled into the hopper of a sprinkling (also known as raining or showering) device. The gross weight was measured by an electric load cell. The sprinkling device was placed over the SELG base and maintained a known distance above the surface (24 in.); while the device was slowly turned at a constant rate, the sand dropped through the hoses, Fig. B.1. A density-height of fall study was made to determine an optimum turning rate and height of

Private communication with J. G. Jackson, Jr., Chief of the Impulse Load Section, Soils Division, U. S. Army Engineer Waterways Experiment Station, CE, Vicksburg, Miss., April 13, 1965.

fall.\* The full merits of the sprinkling technique are discussed by Whitman and others (1962, Appendix B).

The sand was placed up to the proposed level of the bottom of the cylinder. The cylinder was then positioned and sand was sprinkled in a manner intended to duplicate the free-field placement to bed the lower portion (90 degrees to 270 degrees). A piece of cardboard was used to deflect the sand beneath the spring line. The sprinkler was then repositionel and the remainder of the sand placed. The excess was screeded from the top to form a flat surface. A study\*\* to determine the effect of sprinkling sand around a small-scale buried structure has shown that the density in the vicinity of the structure can be about 2 pcf less than the average density in the free field.

The average density was  $106.6 \pm 1.0$  pcf. There was more scatter in average density with the sprinkling technique than with the vibration technique used for the Sangamon River sand.

B.2 Soil Strength and Deformation Characteristics

B.2.1 Sangamon River Sand

This sand was obtained from the Pontiac Stone Company, Mahomet, Illinois. It was wet and not of desired gradation when received. A system outlined by Prakash (1962, p 223) was used to obtain a uniform sand comparable to that tested by Hendron (1963). The sand was spread on the floor of

\* W. J. Turnbull, Chief, Soils Division, WES, "Soil Tests on Sprinkled Cook's Bayou No. 1 Sand Small Blast Load Generator Specimens," Memorandum for: Chief, Nuclear Weapons Effects Division, July 22, 1964.
\*\* R. W. Cunny, Chief. Soil Dynamics Branch, Soils Division, WES, "The Effect of an 8-In.-Dismeter Arch on the Density Produced by Showering Placement Method," Memorandum for: Chief, Muclear Weapons Effects Division, December 1, 1964.

the Illinois civil engineering test track and dried. Then, 8-lb batches were subjected to 5 min of sieving on a Gilson shaker (Model CL-262) that was fitted with a No. 40 and No. 60 sieve. The material retained on the No. 60 sieve was utilized for this investigation. The grain size distribution is shown in Fig. B.2.

The static, stress-deformation characteristics in triaxial and consolidometer tests are presented in Fig. B.3. These tests were run on sand having a density as close as possible to the overall average density used during the cylinder tests. The relative density,  $D_r$ , is also listed in Fig. B.3. Standard procedures were used.\* Moduli and shear strength data are presented in Fig. B.5.

B.2.2 Cook's Bayou Sand

This sand is commonly used in most of the WES blast load generator experiments, e.g. Tener (1964). It was procured locally and its characteristics were originally documented (then called Bayou Pierre Sand No. 1) in a WES Soils Division Memorandum for Record.\*\* However, recent laboratory tests performed for this investigation, Figs. B.4 and B.5, indicate that the one-dimensional stress-strain curve and angle of internal friction for a density of 106 pcf in the memorandum were in error.

It is evident that the two sands used have nearly identical laboratory properties at the densities employed because they were placed at equal relative densities. Also, the differences in the techniques used

\* Described in a laboratory manual prepared by the Waterways Experiment Station for the Office, Chief of Engineers, which will be issued as a Corps of Engineers Engineer Manual.

<sup>\*\*</sup> P. F. Hadala, Impulse Load Section, Soils Division, WES, "Soils Laboratory Tests on Bayou Pierre Sand No. 1," Nemorandum for Record, 1963.

to place the sand in the vicinity of the cylinder negate any refinements in explaining differences in laboratory soil properties. The sand around the cylinder in the Cook's Bayou sand tests may have been only of medium relative density.

# **B.3** Elastic Properties

Hendron (1963, p 84) concluded that the coefficient of earth pressure at rest,  $K_0$ , varies inversely with the angle of internal friction,  $\not o$ , as determined from drained triaxial tests.

$$l_{\mu} = 1 - \sin \beta \qquad B.1$$

For these sands,  $p = 36^\circ$ , Fig. B.5, and therefore

$$K_0 = 2 - \sin 38^\circ = 1 - 0.6 = 0.4$$
 B.2

If the soil were an elastic medium,

$$K_0 = \frac{v_8}{1 - v_8} \qquad B.3$$

and hence

$$v_{\rm g} = \frac{K_{\rm o}}{1+K_{\rm o}} = \frac{0.4}{1.4} = 0.19$$
 B.4

where v is Poisson's ratio for the soil.

Young's modulus of elasticity for the soil,  $\mathcal{B}_{s}$ , may be expressed in terms of the constrained modulus from t e consolidation test,  $M_{c}$ , as

$$E_{s} = \frac{(1 + v_{s})(1 - 2v_{s})}{(1 - v_{s})} M_{s}$$

$$= \frac{(1 + 0.29)(1 - 0.58)}{(1 - 0.29)} M_{s}$$

$$= \frac{(1.29)(0.42)}{0.71} M_{s} = 0.76 M_{s}$$
B.6

Variations of the constrained secant modulus,  $M_{CS}$ , with vertical pressure are plotted in Fig. B.5. Une-dimensional properties obtained at

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several relative densities for the Cook's Bayou No. 1 sand are reported by McNulty (1965).

Whitman and Healy (1962), discussing triaxial test results, and Davisson (1964), discussing one-dimensional test results, have indicated that essentially no dynamic strain-rate effects exist for dense, dry sands of the type used in this investigation.



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Fig. B.2 Gradation Curves for the Sands

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Fig. B.3 Stress-Strain Relations for Sangamon River Sand



Fig. B.4 Stress-Strain Relations for Cook's Bayou No. 1 Sand

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#### APPENDIX C. PROPERTIES OF BUCKSHOT CLAY

### C.1 Placement Techniques

The placement of small-scale test structures in a clay soil is a new endeavor. Luscher and Höeg (1964, pp 219-225) pointed out some of the difficulties in soil control. Inherent in the WES test setup are two additional difficulties: (a) the cylinder ends are closed before burial so that strutting the diameters is impracticable; (b) the cylinder cannot be positioned vertically for soil placement and then positioned horizontally for loading because the test chamber is a complete ring and cannot be sectioned.

It was therefore decided to use a technique already developed for footing tests, Carroll (1963) and Jackson and Hadala (1964), to place and compact the clay to the top of the 2-ft-deep test device. The cylinder would then be placed in the medium by cutting out a trench of appropriate dimensions in the center of the clay specimens and carefully backfilling around the cylinder. The technique for accomplishing the latter task was determined and patterned after procedures described in a feasibility study.\* Although adequate for the present investigation, the technique still has some drawbacks which will be discussed below.

The procedure followed in placing the 2-ft-thick clay specimen in the SBLG ring is shown in Fig. C.1. The clay was mixed in a pugmill and brought to the test area by truck, Fig. C.1a. When stored, the clay was kept continuously sealed in a polyethylene membrane (wrapper) except when

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<sup>\*</sup> R. W. Cunny, Chief, Soil Dynamics Branch, Soils Division, WES, "Tentative Placement Technique for Cylinders Buried in Clay Specimens," Memorandum for: Chief, Nuclear Weapons Effects Division, 1965 (in preparation).

soil was removed. The soil was processed on Fridays and allowed to cure over a weekend. The soil was weighed so that each loose lift, Fig. C.lb, would produce a 2-in. compacted lift. The loose soil was first hand-tamped, Fig. C.lc, and then compacted by three passes of a pneumatic tamper, Fig. C.ld. The soil surface was scarified, Fig. C.le, between lifts. The quality of the placement was controlled and chercked primarily through the use of density samples, Fig. C.lf. Vane-shear tests, Fig. C.lg, were made for certain specimens. Unconfined and confined compressive tests were performed on soil cubes that were taken from the top 8 in. of the specimen before and after each test. The pretest cube was taken at a distance of 1 ft from the cylinder and the hole was filled in a manner to duplicate the free-field placement. These results, as well as water content and density determinations, are given in Table C.l.

The cylinder was placed by cutting out an area in the center of the 2-ft-thick clay specimen, Fig. C.2a. The length and width of the cavity were the same for all tests and only the depth was varied. A template was used to size the excavation, Fig. C.2b, and it also served as a guide for the scooping operations, Fig. C.2c, which cut out a seat for the bottom half of the cylinder. The half-cylindrical cavity was formed exactly to the cylinder dimensions, and areas were carved out to accommodate the strain gages and end nuts, Fig. C.2d. The cylinder was then placed in the carved-out area, Fig. C.2e. The backfill was placed in loose, 3/4-in. lifts, Fig. C.2f, and compacted by three passes of a Harvard miniature compactor, Fig. C.2g. A lift is shown in place in Fig. C.2h. It is believed that very close contact was achieved between the clay and the cylinder.

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All 16 hoop strain gages were monitored during the placement operation. Some strain was impressed into the cylinder during each phase of the placement. Although several remedial methods (such as imposing a small vertical load on the cylinder through a saddle adapter) were tried to eliminate the strains, it was only possible to minimize them. About 40 percent of the total strain caused by placement occurred during the first seating phase, Fig. C.2e. Much of the remainder came during the first and second backfilling lifts; very little disturbance was noted in the cylinder due to lifts placed after the crown was covered.

The strains were primarily bending in nature and were most severe at the quarter points. The strains indicated that the cylinders assumed a slight vertical-elliptical shape. They probably did not significantly influence the failure pressure or mode of failure.

The average impressed strain resulting from the placement is shown in Fig. C.3. It is apparent that this placement technique must be improved before it can be applied to more flexible cylinders. Dorris and Albritton (1965) had very satisfactory results with this technique on a stiffer cylinder ( $EI/R^3 = 82$ ).

The placement technique was tedious and required a considerable amount of time. So much hand labor is involved that each of the ten tests required an average of one week in the testing device. Great care had to be taken to keep the clay sealed to avoid moisture loss. The water content determination from the cube tests indicates that this was successful (Table C.1).

The placement technique in the WES laboratory is probably better than that which could be achieved in a field installation.

# C.2 Soil Strength and Deformation Characteristics

The gradation curve and specific gravity,  $G_s$ , are shown in Fig. C.4. The clay is classified as a CH, and the results of several Atterberg limits tests are shown in Fig. C.5. The static, unconfined compressive strengths,  $q_u$ , were determined in the laboratory from samples taken from 8-in. cubes cut from the in-place clay specimens (the hole was refilled to the same density). The results are plotted in Fig. C.6. Average values are listed for each test in Table C.1.

In order to establish the quality of the backfill, specimens of clay were compacted in a mold in as nearly the same manner as the backfill was compacted. Unconfined specimens were cut from the mold and tested. The results are plotted in Fig. C.6. Those, coupled with the information from the vane-shear tests, indicate that the backfill was about 25 percent weaker than the compacted soil in the free field. Some of the weaker mold specimens were honeycombed (visual inspection) and this resulted in the lower values plotted in Fig. C.6 and the lower density values plotted in Fig. C.9. These molds were made during the early weeks of the investigation, and they may not have been truly representative of the compacted backfill.

Static triaxial (UU) test results are plotted in Fig. C.7. The degree of saturation,  $S_r$ , was about 90 percent and an apparent friction angle,  $\emptyset$ , equal to 1.7 degrees was deduced.

In order to establish an approximation to the one-dimensional stress-strain relation, three consolidation tests were run in which the vertical stress was applied and the deformation recorded as a function of time, Fig. C.7.

Moduli from the triaxial (UU) tests are plotted against confining pressure,  $\sigma_3$ , in Fig. C.8. These moduli exhibited a negligible increase with confining pressure and a line representing the average value is shown. Also in Fig. C.8, the secant modulus from the consolidation test at 6 sec elapsed time (after load application) is plotted with respect to vertical pressure.

The dry density,  $\gamma_{\rm d}$ , is shown in Fig. C.9 with respect to water content. It can be seen that the in-place soil is very similar to that used by Jackson and Hadala (1964).

Carroll (1963) conducted dynamic triaxial tests on buckshot clay (w = 27.1%) and indicated that the clay is strain-rate sensitive. However, the dynamic cylinder tests of this investigation either masked the effect or did not benefit from it. Kane and others (1964) discussed the behavior of clay under rapid and dynamic loading.

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Table	

Pretest Properties of Clay Specimens

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									Average Dry	e Dry						
									Umit Weights,	t s. pcf	Average UC Compres-	e uc				Average
			Avernge	e Water	Content	t, w, Perc	cent	Cube Tests,	đor		sive	ą	Ave	Average	Pressure	Posttest
	ő		Requested	At	-	From Top 8 in.		Degree of Satura-	8 tn.	Soil Cube	Strength tons/so f	leth Pt	Vane Streng tons/an ft	Vane Strength tona/so ft	Applied During	UC Com- Duressive
Test	S a	Construction Ring Date	From Pug In Density Cu Pug Mill Mill Truck Samples Te	<b>B</b>	In Truck	Density Samples	be ste	tion, S <sub>r</sub>	Ring	Tests Yd	Soll Cube Mold		Soil Field	Backfill	Cyl. Test psi	Strength tons/sq ft
D-1	ч		26	25.2	2h.9	24.9	2.2	6•68	96.3	96.0	2.90	1.65	3.97	ł	300	3.8
D-2	ß	17/30/64	25	24.7		24.1	24.2	91.3	97.3	96.1	3.16	1	ł	ł	130	3.6
D-3	m	<del>1</del> 9/1/21	25	24.7		23.8	23.9	87 <b>.</b> 2	97.8	6*96	2 <b>.</b> 83	ł	4 1	:	100	h.8
7-0	-#	<b>₩9/</b> ₩1/21	25	25.2		24.5	24.4	88.9	96.5	96.7	2.72	2.22	ţ	:	190	3.9
D-5	ŝ	<del>1</del> 9/12/21	25	25.2		24.3	24.3	90•2	9.96	97.5	3.38	1.89	8	ţ	180	ग्"ग
D-10	9	1/11/65	25	25.5		24.1	24.0	88 <b>.</b> 5	98.1	97.3	3.05	ł	1	1 1	91	3 <b>.</b> 4
6-a	7	1/18/65	25	25.5		23.7	23.7	89.1	3 <b>.</b> 6	96.1	3•35	1.91	5.28	4.22	148	3.6
n-8	80	1/25/65	25	27.0	25.0	25.6	25.0	88.2	95.3	95 <b>.</b> 4	2.52	3.10	4.20	ł, 06	911	2.9
D-7	6	2/1/65	25	27.0		26.1	26.7	90.7	95.0	93.8	1.50	2.18	:	1	180	2.1
<b>D-</b> 6	10	2/8/65	25	;	ł	24 <b>.</b> 6	24.3	0*68	6•96	6•96	3,06	2.76	1	1	160	3.1
				_ <	Average		24.6	89.3	96.8	96.7	2.89	2.24	1.48	μ		
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Fig. C.4 Gradation Curve for Buckshot Clay



Fig. C.6 Unconfined Compressive Strength-Water Content Relation





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Fig. C.9 Density-Moisture Content Relation for Buckshot Clay

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#### APPENDIX D. TRANSDUCERS

# D.1 Strain Gages

Gages 3/8 in. long were used because gages long enough to give reasonable average strains but short enough to eliminate the necessity of making curvature corrections were desired.

However, to initiate the investigation while the 3/8-in. gages were being procured, cylinders B-1 through B-5 were instrumented with 1/4-in.-long gages. They were Budd Metalfilm strain gages, Type C12 141B, 1/4 in. by 1/8 in. The remainder of the B group and all of the A and C groups were instrumented with Type C12 161. These gages were all temperature-compensated for aluminum. They are not classified as postyield strain gages but are capable of measuring strains accurately to 4-5 percent, according to the manufacturer. The gages functioned satisfactorily on tension test specimens (Appendix A) in that they measured strains accurately to values greater than 2 percent, and appeared to perform satisfactorily for the cylinder measurements.

Procurement complications prevented the acquisition of identical gages for cylinder groups D and E. Instead, Baldwin-Lima-Hamilton gages, Type FA-37-12-S13, were used. These are also 3/8-in. gages which are temperature-compensated for aluminum. The manufacturer indicates that these are accurate to 2 percent strain and they performed satisfactorily on tension test specimens strained beyond 1-5/10 percent.

Eastman 910 comment was tried as a gage adhesive on several tension test specimens, but was found to be unsatisfactory for strain levels beyond 0.5 percent. Armstrong adhesive C-2 was used to bond

all of the strain gages to the cylinders.

The inside gages were waterproofed by an application of Gagekote No. 1 (a solvent-thinned synthetic resin compound) while the outside gages were covered with Gagekote No. 5 (a two-compound, rubber-like epoxy resin) followed by Gagekote No. 2 (a solvent-thinned nitrile rubber) to isolate them further from the soil media.

A limited study was made to determine the potential influence of the soil pressure (acting as a normal force) on the outside strain gages. Four gages were mounted on a piece of 1/2-in.-thick aluminum plate and covered with various protective coatings, Fig. D.1. Gage 1 had a metal cover so that no soil pressure could reach the gage, and hence it served as a control on the response of the other three gages. All gages were waterproofed. Gage 2 was covered with a 0.015-in.-thick strip of fishpaper, gage 3 with Gagekote No. 5, and gage 4 with a piece of electrical, rubber tape. The plate was horizontally buried in sand and loaded statically to 300 psi. Negligible differences were noted in the response of the four gages, and the technique used for gage 3 was selected for its ease of use.

#### D.2 Diameter Change Gages

A diameter change gage was required which would be expendable since the cylinder collapse would destroy anything inside. The transducer used was recommended by Professor V. J. McDonald of the University of Illinois. It consisted of a curved strip of 0.01-in.-thick brass shim stock 1/4 in. by 6 in., Fig. D.2. Budd Netalfilm, Type Cl2 141B, strain gages were mounted on each side of the strip's center with Eastman 910 cement. The gages were joined electrically to indicate only the bending

strains. Two 1/32-in.-diameter holes were drilled in each end of the strip and in the cylinder. The same nut and bolt arrangement was used for mounting the strip in the cylinder, Fig. 4.1a, as was used in calibration.

Each diameter change gage was calibrated in extension and compression in a Pratt and Whitney Super Micrometer. The apparent strain gage output was a linear function of displacement, and amounted to  $5 \,\mu in./in.$ per 0.001 in. of diameter change.

The gage could not be used for rapid or dynamic testing because it experienced excessive ringing under these loadings. Gages were coated with petroelastic to dampen the spurious vibrations but no improvement resulted.

D.3 Overpressure Gages

For the tests conducted at the University of Illinois, Bourdon gages were used to measure the static overpressure. Their accuracy was verified relative to other available gages.

The rapid pressure tests were monitored by a Kistler piezoelectric pressure transducer Model 601. The transducer was calibrated prior to testing and its output was a linear function of overpressure, 0.41 picocoulombs per psi or about 125 psi per inch of paper deflection. The gage was checked after each test, and no calibration changes were required.

Both the static and dynamic tests at WES were monitored by Norwood pressure transducers Model 211. These were statically calibrated prior to each test, and exhibited a generally linear response. They were ranged for about 250 psi/in. of paper deflection statically, and 125 psi/in. of paper deflection dynamically.

At least two gages were used in each test and the measured pressure for the gages was within ±5 percent of the sverage. A Bourdon gage was used to verify the peak static pressure and thereby made the static values more reliable; but the dynamic results probably varied either because of the use of a static calibration or because of the motions of the gage mounts. The gages were located between the firing tubes in the dynamic bonnet. A study by Kennedy and Sadler (1965) has shown that the surface pressure distribution is uniform within ±10 percent.

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Fig. D.1 Strain Gage Test





Albert Francis Dorris was born in Utica, New York, on October 25, 1936. Following graduation from Utica Free Academy in 1955, he entered the United States Military Academy. He was named to the 1957 All-American Trok and Field Team. In his senior year, he was appointed a Cadet Captain. He received the Bachelor of Science Degree in June 1959 and stood 6th in graduation order of merit cut of a class of 499. Upon graduation he was commissioned as a regular officer in the United States Army, Corps of Engineers. He attended the Engineer Officers' Basic Course at Fort Belvoir, Virginia, and the U.S. Army's Airborne and Ranger courses at Fort Benning, Georgia, enroute to his initial assignment in Korea. From June 1960 to June 1961 he was with the 547th Engineer Company (Float Bridge) in Korea as a Platoon Leader and Company Commander. In August 1961 he entered the Engineer Officers' Advanced Course at Fort Belvoir and subsequently worked for a brief period in the Office of the Chief of Engineers before entering graduate training in Civil Engineering at the University of Illinois in June 1962. He received the Master of Science Degree from Illinois in June 1963 and from June 1964 to June 1965 was assigned to the U.S. Army Engineer Waterways Experiment Station, Vicksburg, Mississippi, as a Projec+ Engineer. He currently holds the rank of Captain in the Corps of Engineers and is a member of the American Society of Civil Engineers, the Society of American Military Engineers, and the Association of the United States Army. He is an Engineer-in-Training in the State of Illinois.

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