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References:

- Milne-Thompson: "Theoretical Hydrodynamics", Macmillan, Chapters 11, 12.
- Kreisel, G.: "Cavitation with Finite Cavitation Numbers", Admiralty Research Laboratory Report.
- Kisbouchinski, D.: "Sur les singularités des mouvements fluides", Proceedings of the Second International Congress for Applied Mechanics, pp. 512-518.

Encls:

- (HW) Figures 1 through 4.
- (HW) Tables I and II.

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It is well known that the flow of a liquid past a sharp corner is characterized by a velocity singularity at the corner. This singularity is a result of the fact that the velocity must be zero at the corner, while it is finite in the rest of the flow. This singularity is a result of the fact that the velocity must be zero at the corner, while it is finite in the rest of the flow. This singularity is a result of the fact that the velocity must be zero at the corner, while it is finite in the rest of the flow.

The extent of cavitation is determined by the cavitation number,  $K$ , which is defined as the ratio of the static pressure to the dynamic pressure. In water tunnel experiments, there are limitations on the maximum attainable velocity  $V$ , the cavitation number is reduced by letting  $P$  approach  $p_c$ ; in water entry experiments, the dynamic head is controllable, so that  $P/\rho V^2$  can be made as small as desired.

$$K = \frac{P - p_c}{\rho V^2}$$

The size of the cavity increases as  $K$  approaches zero, that is, as  $P$  approaches  $p_c$  or as  $P/\rho V^2$  approaches zero. In water tunnel experiments, there are limitations on the maximum attainable velocity  $V$ , the cavitation number is reduced by letting  $P$  approach  $p_c$ ; in water entry experiments, the dynamic head is controllable, so that  $P/\rho V^2$  can be made as small as desired.

3. The theory of the cavity formed when the static pressure in the liquid is equal to the pressure in the cavity is identical in the two-dimensional case with the theory of the infinite wake (Figure 1-a) which was first explored by Kirchhoff and Helmholtz, and later extended by Levi-Civita and others; see reference (a) for some details of this work. Physically the infinite cavity cannot occur, but can be considered as the limiting case of what occurs as the dynamic head increases without limit relative to the static pressure in the undisturbed liquid, or as the static pressure approaches the cavity pressure.

4. The Kirchhoff flow is limited to infinite cavities and zero cavitation number. An exact theory of the finite cavity corresponding to non-zero cavitation number has only recently been given. This theory is attributed to H. Wagner, although unpublished by him, and was developed independently in 1945 by G. Kreisel of the Admiralty Research Laboratory, England (reference 1). The essential feature of the theory is the occurrence of a reverse jet extending from the rear of the cavity through the obstacle to infinity, as shown in Figure 1-b. Although this seems physically objectionable, it is a good description of what has been observed experimentally at the Naval Ordnance Laboratory. Jets do indeed occur in finite cavities, although they are either weakened by the turbulent mixing at the rear of the cavity, or finally collide with the obstacle.

1-1

1-1

1-1

is easily seen from the fact that the velocity potential satisfies the Laplace equation,  $\nabla^2 \phi = 0$ , in the region  $z > 0$ . The boundary conditions are  $\phi = 0$  and  $\partial \phi / \partial z = 0$  at  $z = 0$ . The velocity potential is given by  $\phi = Ux + \frac{1}{2}U^2 \frac{z^2}{U^2}$ .

As a particular example, consider the case of a flow past a flat plate. The flow is irrotational and incompressible. The velocity potential is given by  $\phi = Ux + \frac{1}{2}U^2 \frac{z^2}{U^2}$ . The streamlines are given by  $\psi = Uy + \frac{1}{2}U^2 \frac{z^2}{U^2}$ . The flow is shown in Figure 1-a. The pressure is constant in the region  $z > 0$ . The velocity is given by  $v = U + \frac{1}{2}U^2 \frac{z}{U^2}$ .

$$U + \frac{1}{2}U^2 \frac{z}{U^2} = p + \frac{1}{2}\rho U^2$$

where  $U$  is the (constant) flow velocity on the free streamlines. Thus

$$\frac{U^2}{2} = \frac{P - p}{\rho} + \frac{1}{2}U^2$$

When  $P = p$ , and therefore  $N = 0$ , the classical Kirchhoff type flow shown in Figure 1-a prevails.

8. For the case  $P > p$ , consider the flow shown in Figure 1-b. We shall need only the upper half of the flow plane, which will be designated the  $z$ -plane. The flow problem is solved once the complex velocity potential,  $w(z) = \phi + i\psi$ , is determined, where  $\phi$  is velocity potential,  $\psi$  is stream function.

- \* This is required by the fact that the pressure always increases from the cavity into the flow.
- \*\* The following treatment can very easily be applied to wedges also.

The function  $Q$  is analytic in the  $\zeta$ -plane, except for branch cuts along the real axis from  $\zeta = b$  to  $\zeta = c$  and from  $\zeta = 1$  to  $\zeta = \infty$ .

$$Q(\zeta) = \frac{K_1}{(\zeta - c)^{1/2} (\zeta - 1)^{1/2} (\zeta - b)^{1/2}}$$

The branch cuts are shown in the  $\zeta$ -plane. The function  $Q$  is analytic in the region  $\text{Im } \zeta > 0$ ,  $\text{Re } \zeta > c$ , and  $\text{Re } \zeta < 1$ . The branch cuts are along the real axis from  $\zeta = b$  to  $\zeta = c$  and from  $\zeta = 1$  to  $\zeta = \infty$ .

- 1.  $\zeta = c$
- 2.  $\zeta = 1$
- 3.  $\zeta = b$
- 4.  $\zeta = \infty$

The branch cuts on the  $\zeta$ -plane are shown by the lines marked with arrows.

(1)  $Q$ -plane on the  $\zeta$ -plane

$$\frac{dQ}{d\zeta} = \frac{K_1}{(\zeta - c)^{3/2} (\zeta - 1)^{1/2} (\zeta - b)^{1/2}}$$

with subsidiary conditions obtained from the known jumps in  $Q$  at  $0$  and  $c$ ,

$$K_1 = c \sqrt{c-1} / 2 = c \sqrt{c-1} \sqrt{c-b}$$

(A)  $\therefore \sqrt{c-1} \sqrt{c-b} = \sqrt{c}/2$

(2)  $w$ -plane on the  $\zeta$ -plane

$$\frac{dw}{d\zeta} = K_2 \frac{\zeta - c}{\zeta - b}$$

with subsidiary condition

$$K_2 = \psi_0 / \pi (c - b)$$

From (1) it follows that

$$Q = \frac{1}{2} \log \frac{\sqrt{\zeta - b} + \sqrt{b} \sqrt{\zeta - 1}}{\sqrt{\zeta - b} - \sqrt{b} \sqrt{\zeta - 1}} + \log \frac{\sqrt{\zeta - 1} \sqrt{\zeta - b} + \sqrt{c - b} \sqrt{\zeta - 1}}{\sqrt{\zeta - 1} \sqrt{\zeta - b} - \sqrt{c - b} \sqrt{\zeta - 1}} + \log K_1$$

At  $x = b$ ,  $y = 0$ , therefore  $K_2 = 1$  and

$$\frac{u}{U} = \frac{1}{\sqrt{c-b}} \left[ \frac{c-y}{c-b} \sqrt{\frac{c-y}{c-b}} + \sqrt{\frac{c-y}{c-b}} \right]$$

By taking the limit as  $y \rightarrow 0$ ,  $u = U$ , and  $\frac{u}{U} = 1$ . From (2) it follows that

$$\frac{u}{U} = \frac{1}{\sqrt{c-b}} \left[ \frac{c-y}{c-b} \sqrt{\frac{c-y}{c-b}} + \sqrt{\frac{c-y}{c-b}} \right]$$

The thickness of the jet is  $\frac{4b}{U}$ .

$$\frac{1}{\pi} \int_0^b \frac{4b}{U} \frac{d\psi}{dx} dx = \int_0^b \frac{4b}{U} \frac{d\psi}{dx} dx$$

$$\frac{1}{\pi} \frac{4b}{(c-b)U} \int_0^b \left( \frac{c-y}{c-b} \sqrt{\frac{c-y}{c-b}} + \sqrt{\frac{c-y}{c-b}} \right) \frac{d\psi}{dx} dx$$

The thickness of the jet is  $\frac{4b}{U}$ . The shape of the cavity is given parametrically as a function of  $\psi$  ( $0 \leq \psi \leq b$ ) by the expression

$$\frac{1}{\pi} \frac{4b}{(c-b)U} \int_0^{\psi} \left( \frac{c-b}{\sqrt{c-b}} + \sqrt{\frac{c-b}{c-b}} \right) \frac{1}{2} \left( \frac{c-b}{\sqrt{c-b}} + \sqrt{\frac{c-b}{c-b}} \right) d\psi$$

10. The drag is computed by integrating the excess pressure over the front face of the plate, neglecting the effect of the jet on the back face, however

$$\text{Drag} = C_D \frac{1}{2} \rho U^2 = \rho \int_0^{4b} (U^2 - u^2) dy, \quad (C_D = \text{drag coefficient})$$

$$C_D = \frac{1}{U^2} \left( 1 - \frac{\int_0^{4b} u^2 dy}{\int_0^{4b} U^2 dy} \right)$$

Since

$$\int_0^{4b} u^2 dy = -1 \int_0^{4b} \left| \frac{du}{dy} \right|^2 dx = -1 \int_0^b \frac{du}{dx} \frac{dx}{d\psi} \quad \text{or} \quad = 1 \int_0^b \frac{du}{dx} \frac{dx}{d\psi}$$

then

$$C_D = \frac{1}{U^2} (1 - L_1/L_2) = (1 + M) (1 - L_2/L_2)$$



where

where  $\gamma$  is the integrand in the contour integral,  $\gamma_0$  is the value of integrand in the  $\zeta$ -plane excluding the singularity at  $\zeta = 1$ . The contour of integration is taken as the segment of the real axis in the  $\zeta$ -plane (corresponding to the width of the plate in the  $z$ -plane), plus the branch principal value as the value of the integral, the real part of the result is the distance between stagnation points, while the imaginary part is the thickness of the jet which corresponds to the jump at  $\zeta = 1$ .\*

11. The integrals in the expression for  $C_D$  can be written in terms of elementary functions.  $C_D$  was evaluated for cavitation numbers between 0 and 1.5. The curve of drag coefficient vs. cavitation number is shown in Figure 2, and is seen to be almost a straight line. The variation of  $1/\sqrt{12}$  with cavitation number is shown in Figure 3; over the region considered the total variation remains less than 8%. To a good approximation, it follows

$$C_D = C_{D_0} (1+N) = 0.88(1+N)$$

where  $C_{D_0} = 0.88$  is the drag coefficient of a flat plate with infinite cavity and zero cavitation number, the well-known result of Kirokhoff. Table I shows the variation of the different parameters and integrals.

12. Calculations of the thickness of the jet show it to be essentially constant and equal to 22% of the plate width over the considered region of cavitation numbers.

13. The length of the cavity, which is taken to be the distance between stagnation points, is plotted in units of plate width in Figure

\* The preceding analysis is contained essentially in reference (b).

the cavity is zero, the total force on the cavity is zero, as would be expected from the D'Alembert paradox.

16. The calculation of the flow about two plates connected by a free boundary\* proceeds similarly to the previous calculation. The upper half of the z-plane (i.e., the flow plane) is mapped on the w-plane as shown in Figure 4-b. Corresponding points on the two planes have the same lettering. The mapping of the lower half of the z-plane is shown in Figure 4-c, which, as before, is readily obtained from the conditions that the flow direction is constant on the free boundary, while the flow velocity is constant on the free boundary. The mapping of the z-plane on the w-plane is here achieved directly by means of the Schwarz-Christoffel transformation

$$\frac{dz}{dw} = \frac{K_1}{(w^2 - a^2)(w^2 - b^2)^{1/2}}$$

giving

$$Q = K_1 \log \left[ \frac{\frac{a}{\sqrt{a^2 - b^2}} - \frac{\sqrt{w^2 - b^2} + a}{w + a}}{\frac{a}{\sqrt{a^2 - b^2}} + \frac{\sqrt{w^2 - b^2} + a}{w - a}} \right]$$

The conditions

$$Q = 0 \text{ at } w = 0, \quad Q = -i\pi/2 \text{ at } w = 1$$

give  $K_2 = 0$ ,  $K_1 = -1/2$ , and therefore, since  $Q = \log \frac{dz}{dw}$

\* Note that the distance between the plates for a flow with preassigned cavitation number is one of the unknowns of the problem. Conversely, if the position of the plates is fixed, the cavitation number is unknown.

the velocity of the fluid at the stagnation point is zero.

By substituting the expression for the velocity potential in the above equation and integrating the expression, we obtain

$$(a) \quad \psi = \frac{U}{\pi} \int_0^{\theta} \frac{1 - t^2}{1 + t^2} d\theta - \frac{U}{\pi} \int_0^{\theta} \frac{1 - t^2}{1 + t^2} d\theta$$

where

$$(b) \quad I_1 = - \int_0^{\theta} \frac{1 - t^2}{1 + t^2} d\theta$$

$$(c) \quad I_2 = \int_0^{\theta} \frac{1 - t^2}{1 + t^2} d\theta$$

being given by (a) above. These integrals can be expressed in terms of elliptic integrals.

$$-I_1 = -E(k, \pi/2) - F(k, \pi/2) - (t^2 - 1) \quad (k = \sqrt{t^2 - 1}/t)$$

$$I_2 = t^2 E(k, \pi/2) - F(k, \pi/2) + (t^2 - 1)$$

where E and F are the complete elliptic integrals of the first and second kind, respectively.

17. The length of the cavity, which is the distance between stagnation points A, A' is given by

$$= \frac{2}{\pi} \frac{\int_0^{\theta} I_1 d\theta'}{\int_0^{\theta} I_2 d\theta'}$$

If the upper integral is replaced by  $\int_0^{\theta} I_2 d\theta'$ , the real part of the resulting expression represents the half-distance between stagnation

... of the cavity length, and the exact value of the cavitation number is taken into account.

The evaluation of the cavity length shows a similar agreement between the two theories. The cavity lengths and elevations are tabulated in Table 11 in units of the plate width, and the former is plotted as a function of cavitation number in Figure 8, where it is compared with the cavity length calculated from the Wagner model. The agreement between the two theories is seen here also to be remarkably close, although detailed information is given on high cavitation number flow cavitation and its low speed cavity length (in units of plate width) in Table 11.

The numerical agreement between these apparently dissimilar theories is not surprising, since the only significant physical difference between the two occurs at the rear portion of the cavity, where it would not be expected to affect the drag, cavity length, or other features of the flow in the large. The finite cavities observed in water-entry or water tunnel studies differ from the two models studied here chiefly in the situation at the rear of the cavity. During the separation of the converging streams of liquid at the rear stagnation point, there is a much weakened jet, or none at all, so that the physical situation is intermediate in shape between the two theoretical models. Although measurements have not been made on two-dimensional cavities to confirm the results presented here, water tunnel studies on the flow about a flat disk, in which the pressures over the face of the disk were integrated to obtain the drag, show a similar linear variation of drag coefficient. This would seem to confirm the results presented here for the corresponding two-dimensional flow.

21. Mathematical solutions do not yet exist for any three-dimensional cavity flows. The applicability of analytic methods seems to be unlikely at present. The close agreement between the simplified Riabouchinski model of the cavity and the basically correct Wagner model suggests that the former will give an accurate description of the three-dimensional cavity flow, and at the same time this model offers some

hope of successful memorial treatment. A check against the adequacy of the theory can also be obtained from already available experimental results. From the theoretical point of view, it may be informative to see the frequency of the various methods of research methods.

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FIG. 9

W-Plane  
 $W = \frac{z}{1+z}$



FIG. 10

Q-Plane  
 $Q = \log \frac{U}{V} + j\theta$

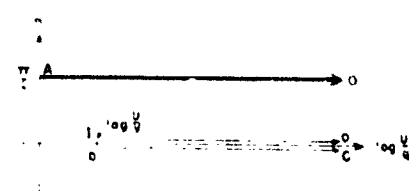


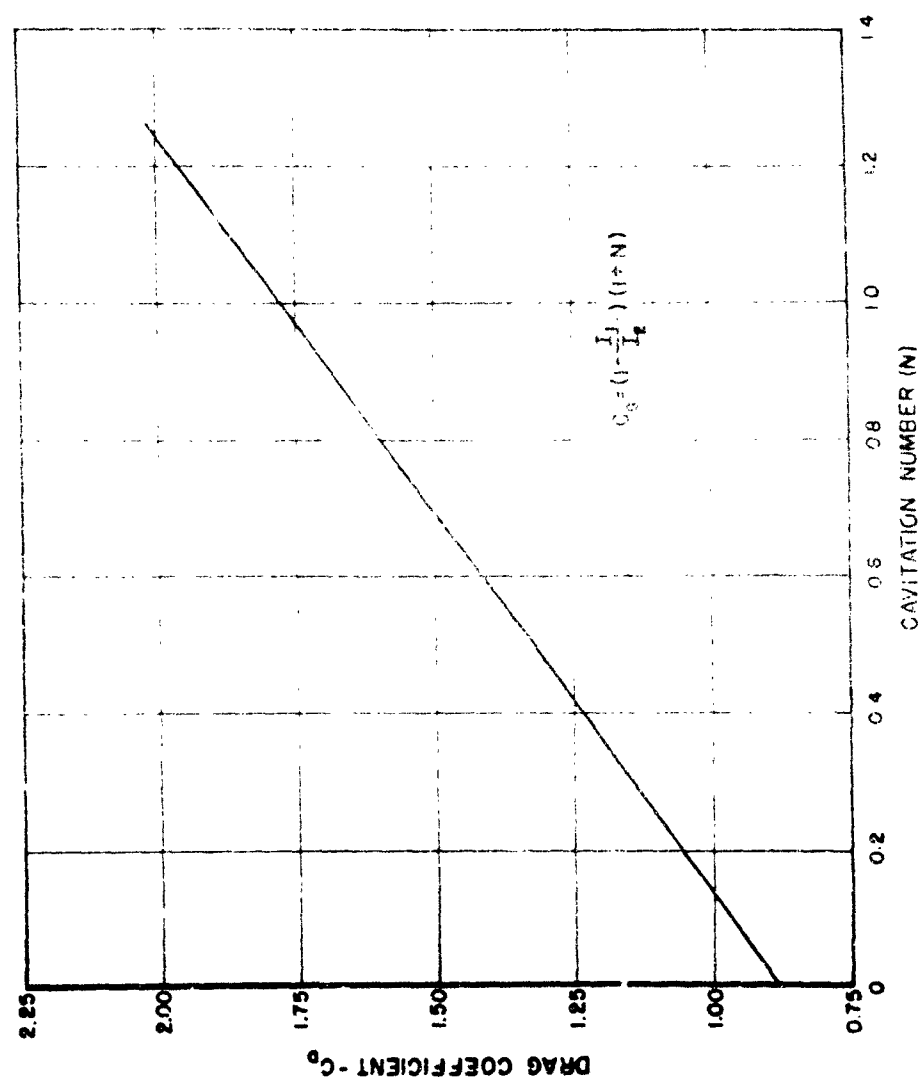
FIG. 11

\xi-Plane



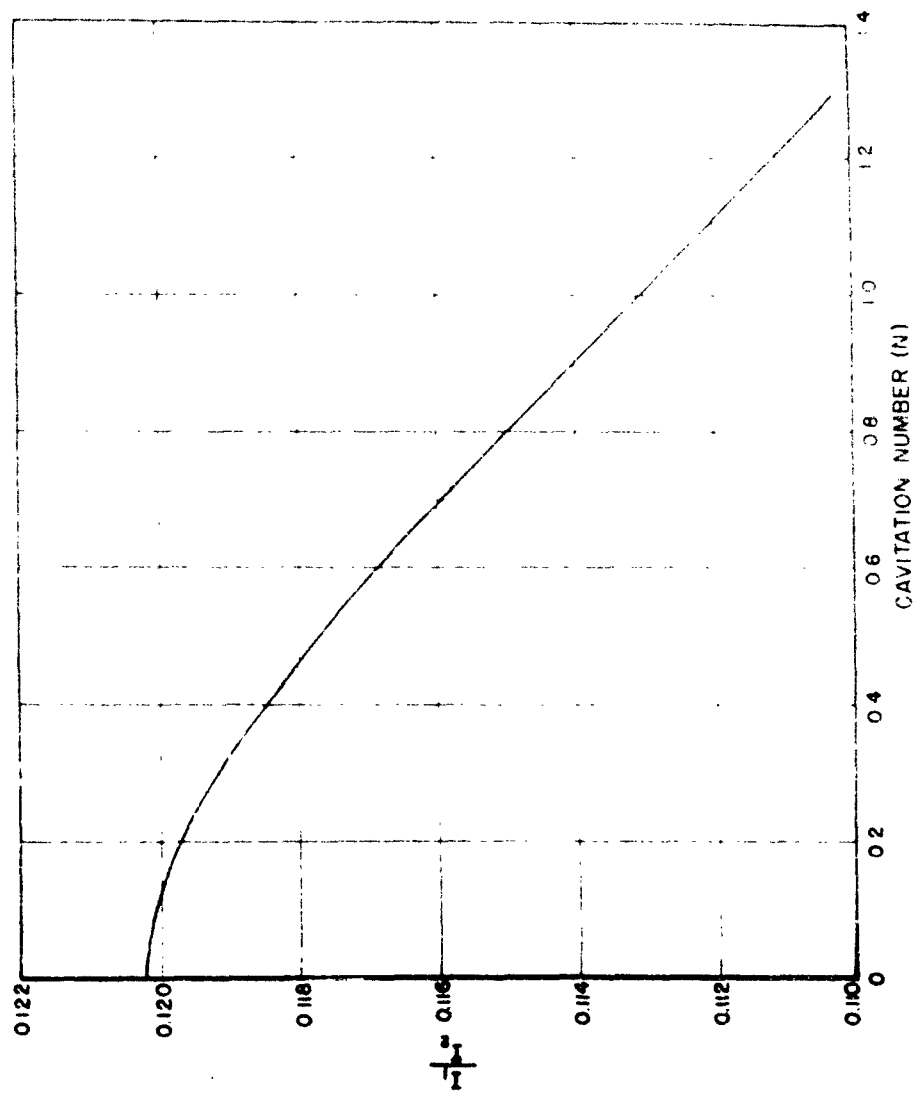
FIG. 12

NSL 8710



DRAG COEFFICIENT AS FUNCTION OF CAVITATION NUMBER

HOLM 578



VARIATION OF  $I_1/I_2$  WITH CAVITATION NUMBER

NOLM 876



Z-Plane



FIG. 4A

W-Plane  
 $W = z^2 + 1$

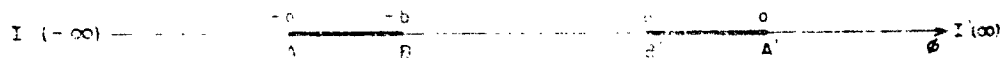


FIG. 4B

Q-Plane  
 $Q = \log \frac{U}{V} + i\theta$

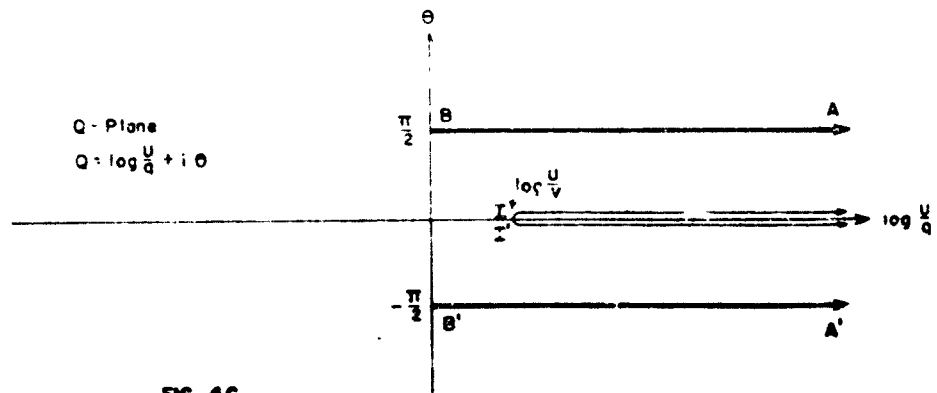


FIG. 4C

Table I

Numerical Values for the Warner Model

$\mu$	$\nu$	$\alpha$	$\beta$	$\gamma$	$\delta$	$\epsilon$	$\zeta$
0.2500							
0.2501	0.4170	3.574	0.2180	0.2195			
0.2504	0.4287	3.582	0.2187	0.2198			
0.2508	0.4281	3.594	0.2191	0.2192			
0.2513	0.4275	3.608	0.2185	0.2185			
0.2519	0.4267	3.624	0.2177	0.2183			
0.2553	0.4222	3.723	0.2154	0.2154			
0.2577	0.4191	3.795	0.2130	0.2133			

Table II

Numerical Values for the Riazouchinski Model

$N$	$11 \times 10^6$	$11 \times 10^6$	$I_0/I_2$	$\mu$	$\nu$	$\epsilon$	$\zeta$
					orvity length	orvity diameter	
0.094	4.293	35.72	0.1202	1.001	279.1	12.97	
0.196	17.14	143.2	0.1197	1.004	69.18	6.72	
0.308	38.48	323.1	0.1191	1.009	30.41	4.64	
0.397	59.74	504.2	0.1185	1.014	19.36	3.82	
0.491	85.14	722.9	0.1178	1.020	13.41	3.28	
0.604	118.8	1016.9	0.1168	1.028	9.45	2.85	
0.695	148.1	1276.4	0.1160	1.035	7.48	2.62	
0.794	181.9	1575.6	0.1151	1.043	6.02	2.49	
0.901	218.5	1915.4	0.1141	1.052	4.92	2.24	
1.004	255.4	2299.0	0.1131	1.061	4.15	2.11	