FOREIGN TECHNOLOGY DIVISION

MANUFACTURE OF PARTS USING THE METHOD OF HYDROSTATIC EXTRUSION

by

A. A. Galkin, V. P. Buryak,
Ye. I. Ozyka, and M. Lisovskiy, et.al.

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The authors study equipment designed and used for the hydroextrusion of industrial shaped products. The new unit operated at 30,000 bar as opposed to the older unit studied by the authors which operated at 20,000 bar. It consists of an attachment incorporating a container coupled to a pressor generator by a plunger. A P477A 100 ton force vertical hydraulic press or a D0437 500 ton force press can serve as the pressor generator. Gasoline, gasoline and oil, kerosene and oil, and various mixtures of these were used as the working fluid with the inclusion of reducing additives. MoS\textsubscript{2} was used for lubrication with various modifications. The optimum entrance cone of the female die was 20 plus 40 degrees. The billet was pressed to an angle equal to that of the female die cone. Circular, rectangular, and spiral industrial shaped products were extruded from nonferrous metals, construction and tool grades of steel, and from hard to form alloys. The dimensions of these products were limited by the available equipment, but did not exceed 12 mm diameter and 150 mm in length. The results show that extrusion requires more force, requiring equipment which can operate at higher pressures.
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MANUFACTURE OF PARTS USING THE METHOD OF HYDROSTATIC EXTRUSION

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Along with the widespread use of methods of producing profiled parts from nonferrous and ferrous metals as well as alloys, interest has developed in a new method of working metals by pressure — the method of hydroextrusion. It is characterized by improved methods of extruding the part through the eye of the matrix. Hydroextrusion is the only rational method of working when manufacturing profiles from materials which are difficult to deform.

The system for extruding profiled parts using fluid under high pressure is the same as the methods for extruding rod-shaped parts, described in the literature [1-4]. The effect of pressure on the mechanical properties of the deformed materials is described in [1,3].

The present article describes an apparatus developed and used for hydroextrusion of profiled parts.

The description does not include the apparatus used in the early stages of the studies, operating at pressures of up to 20,000 bars. The apparatus for hydroextrusion at pressures up to 30,000 bars (Figure 1)
consists of a press attachment, made up of a container 1, connected by a plunger 2 to the pressure generator. The pressure generator is a vertical hydraulic press with a force of 100 tons (Type D 0437).

The pressure in the container is related to the pressure in the cylinder of the press by the relationship

$$p_c = p_k k_f,$$

where $k_m$ is the coefficient of multiplication; $k_f$ is a coefficient which accounts for the losses in pressure transmission; its value depends on the type of compression of the plunger, the working pressure and the fluids and lubricants used; $k_f = 8$ to $20\%$. The apparatus is mounted on two plates 3 and 4, connected to the upper and lower braces of the press. The container is mounted on the spherical surface of a bearing 7 and attached to the plate by a clamp 6.

The working fluid is benzine + oil, kerosene + oil and various mixtures with additives, reducing friction and producing comparatively low extrusion forces. The fluid is poured in at the top with the plunger completely withdrawn.

To reduce friction and cut down the extrusion forces, lubricants are employed: MoS$_2$ in various modifications, hypoid, graphite; lead, zinc and cadmium coatings.

The optimum angle for the inlet cone of the matrix 5 was $20 + 40^\circ$. The blank had a sharpening angle equal to the angle of the cone of the matrix, while a shaft projecting into the calibrating portion with a small clearance was provided for more reliable sealing and accurate centering along the axis of the matrix.
The hydroextrusion process takes place at great speed. With the speed of the press not regulated, it is impossible to regulate the flow process, so that there is difficulty in collecting the parts. This is particularly evident if the volume of the liquid is considerably in excess of the volume of the blank. However, if the volume of the liquid is comparable to the volume of the blank, then as the parts emerge from the press, it will not be possible to maintain the pressure necessary for extrusion, and the process will cease until the pressure builds up again.

To reduce the influence of the accumulated energy of the compressed fluid on the rate of output of the extrudate and the hydraulic system of the press, tests were made involving closing the internal channel of the container following emergence of the part. As a result of the tests, it was concluded that disk-shaped irregular, spherical and conical sections have a positive effect on the operation of the press, insignificantly reducing the rate of output of the extrudate, but reduce the service life of the matrix, especially at high pressures, when the energy of the impact of the cross section against the matrix increases considerably. To collect the parts emerging from the container at high speed, catchers were used with rubber, cloth or other liners, partly solving the problem of braking. Pneumatic and hydraulic catchers were tested.

This device was used for the extrusion of parts with circular, rectangular and helical cross sections from nonferrous metals, structural and tool steels and alloys that were difficult to deform. The dimensions of the parts produced were limited by the available containers and amounted to a maximum of 12 mm in diameter and 150 mm in length.

The container is a single-layer thin-walled cylinder with a ratio of the radii $R/r = 8:10$, autofrettaged by hydrostatic pressure in excess of the working pressure by 15% and working in an elastic-plastic range. The calculations for the strength of thin-walled vessels are discussed in detail in [5, 7].

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Measurements made of the internal channel of the container following 15 cycles of stress up to 30,000 bars showed that no further increase in the diameter had taken place. This is supported by the results of the work in [6], indicating that plastic deformations are damped with an increase in the number of stresses. Vessels are also used in which the plastic deformation amounts to 4-6% (Δr/r·100).

From the experience gained in working with multilayer and single-layer containers, we can conclude that of all the high-pressure containers of current design that operate at pressures up to 30,000 bars, it is best to use single-layer ones.

Experiments were conducted to study the influence of thermal working of the material on the strength of the container. An optimum regime for thermal working of 45 KhNVA, 45 KhNMFA and 45 KhNVFA steels was worked out; these steels are used for manufacture of single-layer containers, used at room temperature. Thermal working up to hardness HRC 39-42 insures high strength of the container.

For working at 500°C, one can use highly alloyed heat-resistant steels (Kh4V2FM, 3 Kh2N2MVF, 3 Kh2V8F, E1958 and Kh6V3FM). The container operates in the elastic-plastic zone, and its working channel has a curved shape, so that a gasket seal was chosen.

In this case, there is no need for a precise working of the internal channel of the container or for fitting it with a working rod; the stress on the rod during longitudinal bending is more favorable; the gasket seal has a long service life.

There are many types of gasket seals. The majority of them consist of a large number of rings, requiring precise working and careful fitting. We tested several varieties of gasket seals and studied the effect of a gap between the moving parts on the mass of the rings.

The simplest seal consists of five rings (Figure 2). Clamping ring 1 is made with a high degree of precision from ShKh15 steel.
The clamping ring forms a gap of 0.01-0.02 mm with the plunger, and 0.015-0.025 mm in the socket. Packing rings 2, 3 and support 5 are made of HRB2 bronze (HRC 60-65).

Teflon is used for sealing ring 4. The ring is made with a clearance of 0.1-0.2 mm relative to the plunger and the socket. The surface life of the packing is determined by the strength of ring 5 and is in the hundreds of cycles.

Type ShKh15, KhVG and R18 steels are used to make the plunger; in working with pressures above 25,000 bars, it is preferable to use R18 steel. At 700°C, use of R12 and 7Kh4V7FM steel may be recommended.

In extruding profiled parts, certain difficulties may be encountered in preparing the matrix. Matrices made of ShKh15, 3Kh2V6F, E1958 and other high-strength steels, as well as refractory steels, are used. Matrices used to make parts with circular and rectangular cross sections are made by a mechanical method, while profiled matrices are made by the electroerosion method or by stamping from cermet alloys.

Refractory matrices are more advantageous, since they are simple to make, have long service lives and provide the required quality of the surface of the emerging extrudate.

The sealing of matrices with a cylindrical outer surface is shown in Figure 3. This kind of sealing is also used for matrix 3 with a nonworked surface and may be employed at pressures up to 30,000 bars. Support ring 1, used for sealing, is made of bronze or brass with a
clearance of 0.01 relative to the matrix and the container. Packing ring 2 (Teflon) is made with a clearance of 0.2 mm relative to the diameters.

RESULTS

Pressing profiles requires high forces, thus creating the necessity for an apparatus working at higher pressures. The use of high-strength ductile steels with a special preliminary hydraulic working of the internal channel makes it possible to use a container with a single-layer construction at room temperature for pressures up to 40,000 bars.

Containers with large diameters for the internal channel for pressures up to 40,000 bars may be made with two-layer construction, made up of two inserts with a slight gap.

It is necessary to continue work in the area of reducing the rate of output of extrudate and in working out designs for capturing the latter and slowing it down in ordinary industrial hydraulic presses. In addition, it is necessary to develop a design for compacting the rod and matrix, insuring a long continuous service at a pressure of 30,000 bars and a temperature of 500° C.
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